

N67 25899

FACILITY FORM 602

(ACCESSION NUMBER)

(THRU)

(PAGES)

(CODE)

(NASA CR OR TMX OR AD NUMBER)

(CATEGORY)

12 116 170

CR-83919

7

31

TECHNICAL REPORT
HREC/5485-1
LMSC/HREC A712573

LOCKHEED MISSILES & SPACE COMPANY
HUNTSVILLE RESEARCH & ENGINEERING CENTER
HUNTSVILLE RESEARCH PARK
4800 BRADFORD DRIVE, HUNTSVILLE, ALABAMA

AUTOMATED NOSE FAIRING
DESIGN -- HONEYCOMB
SANDWICH CONSTRUCTION

Contract NAS8-15485

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9 November 1965

FOREWORD

This Technical Report describes one of three computer programs which were developed as tools for generating parametric weight and design data for nose fairings suitable for Saturn-class payloads. The work was performed by Lockheed Missiles & Space Company, Huntsville Research & Engineering Center, with support from the LMSC/Palo Alto Research Laboratories, for the National Aeronautics and Space Administration/ Marshall Space Flight Center under Contract NAS8-15485, from July through November 1965.

The three computer programs developed under this contract are described in the following three reports.

1. Automated Nose Fairing Design -- Ring and Skin Construction, LMSC Technical Report LMSC/HREC A712552, November 1965.
2. Automated Nose Fairing Design -- Ring, Skin and Stringer Construction, LMSC Technical Report LMSC/HREC A712572, November 1965.
3. Automated Nose Fairing Design -- Honeycomb Sandwich Construction, LMSC Technical Report LMSC/HREC A712573, November 1965.

Many of the subroutines and the methods of specifying external geometry and aerodynamics loads are common to all three programs.

This report describes the computer program for honeycomb sandwich construction. Major contributors to the development of this computer program are B. O. Almroth of the Palo Alto Research Laboratories and E. S. Hendrix, I. M. Landis, and Z. Adams of Huntsville Research & Engineering Center. Appendix K of this report was written by B. O. Almroth, and the remainder was written by Z. Adams.

SUMMARY

The computer program described in this report synthesizes near-optimum designs for honeycomb sandwich nose fairings with an external geometry consisting of up to ten intersecting conical frustums (or cylinders) capped with a spherical nose cap. The combined effects of bending moments, axial loading due to drag, and lateral pressure are considered in performing the design. Either standard gauges or non-standard gauges can be used for faces and ribbons.

The main part of the report is devoted to description of the logic followed in designing a fairing, description of the computer program, and instructions on operation of the program. The program listing and details of the methods of analysis used in design of fairing appear in the Appendixes.

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INTRODUCTION

The computer program described in this report synthesizes near-optimum structural designs for ring stiffened, honeycomb sandwich nose fairings. When the external geometry, aerodynamic loading, ring spacing, and a practical set of design constraints are given, the computer program selects a combination of honeycomb panel face thickness, core height and ring cross-section that gives a minimum fairing weight. Provisions have been made to use either standard or non-standard gauges of material for faces and ribbons. The external geometry can consist of up to ten intersecting conical frustums capped with a spherical nose cap. Figure 1 illustrates the external geometry and type of construction.

A condensed flow chart illustrating the major logical steps performed by the program is shown in Figure 2. Design begins at the base of the fairing and moves toward the nose cap. Each bay is designed to withstand loads imposed by the interaction of bending moments, axial loading, lateral pressure and/or internal pressure. The combination of face thickness, core height, ribbon thickness, cell size and ring cross-section which results in minimum weight-to-volume ratio for a bay is considered to be the optimum design for that bay.

The computer program consists of the main program and a number of subroutines. The logical steps needed to design the fairing are performed in the main program. Specialized and repetitive functions are performed in the subroutines. This particular arrangement, as well as the liberal use of comment cards, is intended to facilitate future modifications.

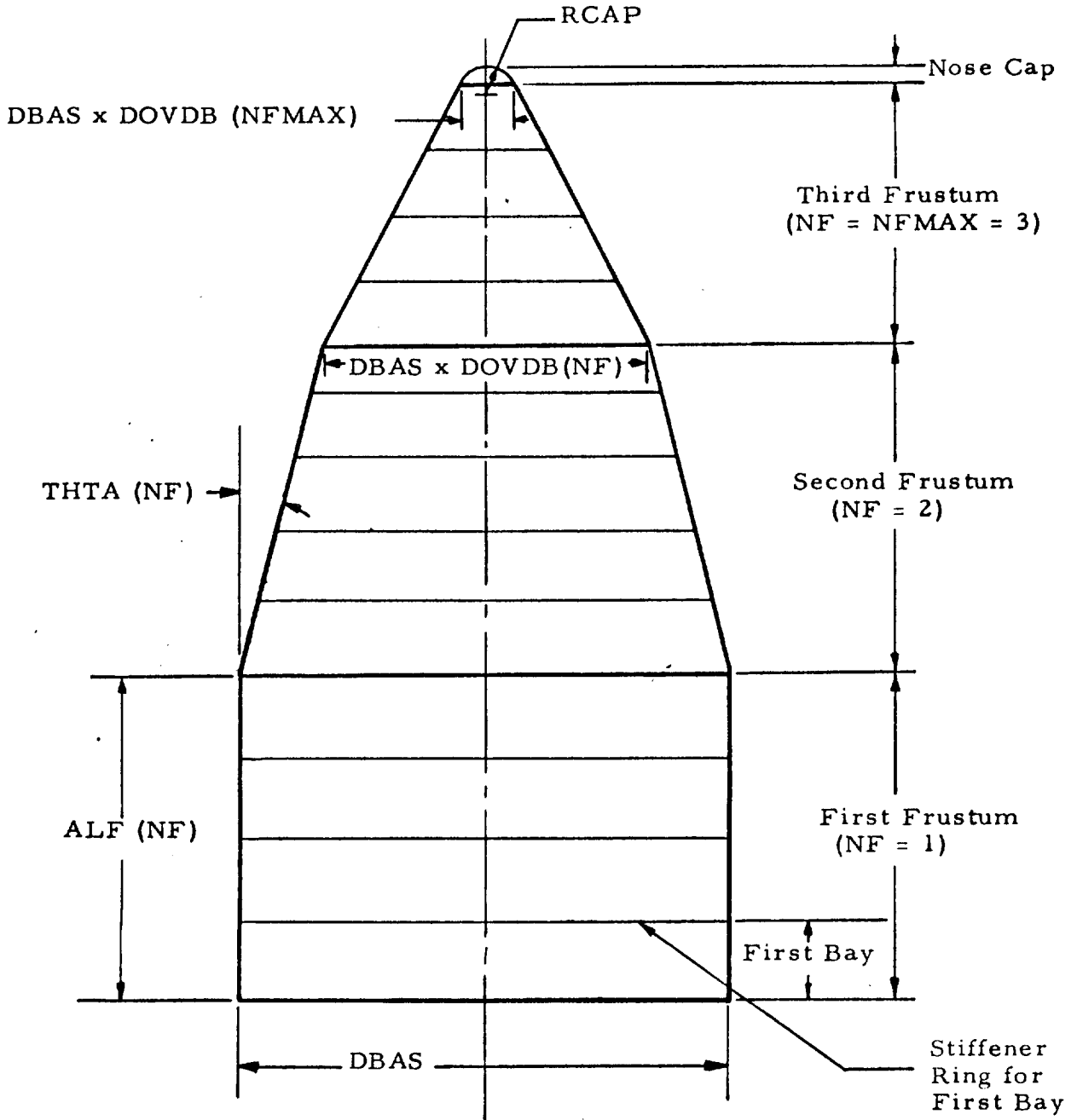
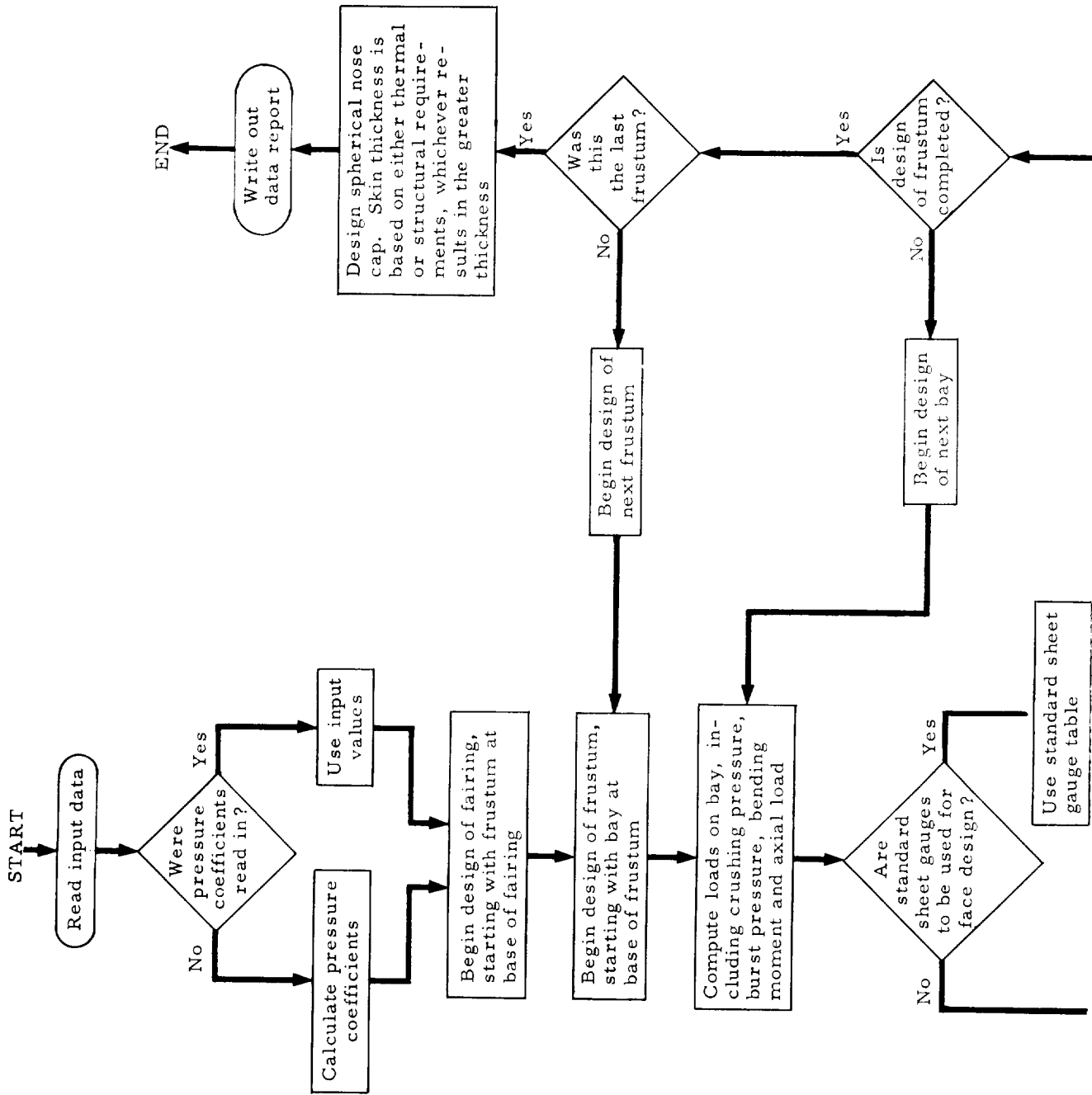
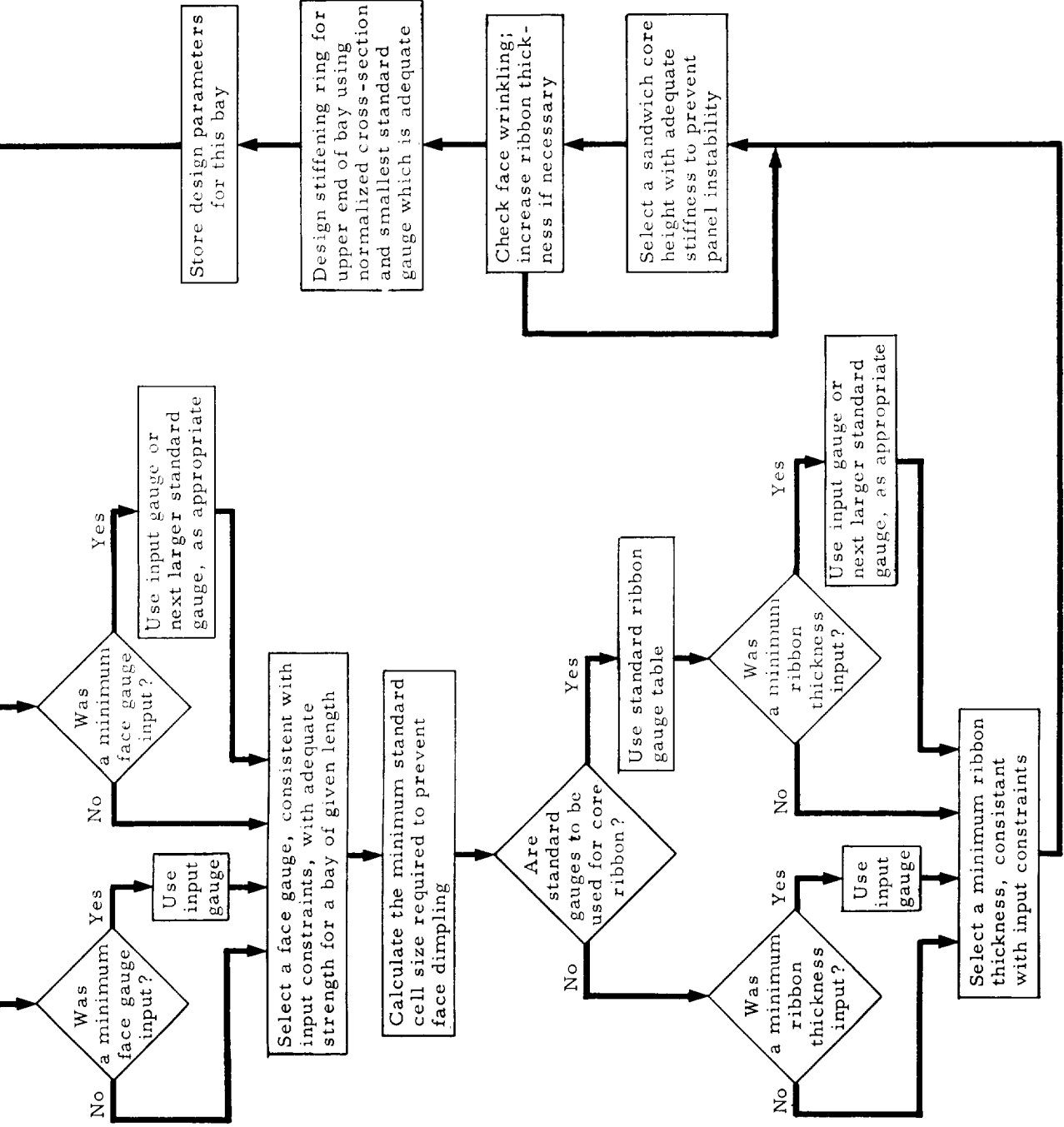


Figure 1 - Nose Fairing Geometry





20, 20

Figure 2 - Flow Chart Showing the Major Steps Performed in Optimizing the Design of a Multi-Frustum Nose Fairing - Honeycomb Sandwich Construction

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C      AXIS. IF THE CELL HEIGHT TO WIDTH RATIO IS LESS THAN 5.0.
C      APPLY A WARPAGE CORRECTION FACTOR BASED ON FIGURE 6 OF THE
C      PENZIEN AND DIDRIKKON PAPER.
      PHI = 0.5 * PI
      GC2 = (GCORE * TRIBN * SBETA * (R + CBETA)) / (A * ((1. + R)
1 * SBETA**2 * (COS(PHI))**2) + (R + CBETA)**2 * (SIN(PHI))**2))
      PHI = 0.0
      GC1 = (GCORE * TRIBN * SBETA * (R + CBETA)) / (A * ((1. + R)
1 * SBETA**2 * (COS(PHI))**2) + (R + CBETA)**2 * (SIN(PHI))**2))
      KY = 0
      H = MAKE FIRST ESTIMATE OF CORE HEIGHT.
      = .25 + TF
      IF (1.GT.1) KY = 1
      T1 = TF
      T2 = TF
251 CONTINUE
      IF ( H/W .GT. 5.0) GO TO 228
      Y = (9.3 / (H/W)) - (2.1 / (H/W)**2)
      IF ( H/W .LT. 0.5) Y = 11.0
228 CONTINUE
      G1 = GC1 * (1.0 + .01 * Y)
      G2 = GC2 * (1.0 + .01 * Y)
      P = PDESMX
      CHECK PANEL STABILITY ON WINDWARD SIDE. IF DESIGN IS
C      SATISFACTORY, PROCEED TO CHECK LEEWARD SIDE. IF DESIGN IS
C      UNSATISFACTORY, INCREASE CORE HEIGHT. IF A HEIGHT OF 4 INCHES
C      IS REACHED AND THE DESIGN IS STILL UNSATISFACTORY, INCREASE
C      FACE THICKNESS.
      CALL INSTBL
      IF (KG .EQ. 1) GO TO 98
      IF (CRG .GE. ANFIMN) GO TO 252
      IF ( H .GT. 2.0) GO TO 229
      H = H + .125
      KY = 1
      GO TO 251
229 CONTINUE
      IF (KTFSTD .EQ. 1) GO TO 234
      TF = 1.1 * TF
      GO TO 239

```

SWD0347
SWD0348
SWD0349
SWD0350
SWD0351
SWD0352
SWD0353
SWD0354
SWD0355
SWD0356
SWD0357
SWD0358
SWD0359
SWD0360
SWD0361
SWD0362
SWD0363
SWD0364
SWD0365
SWD0366
SWD0367
SWD0368
SWD0369
SWD0370
SWD0371
SWD0372
SWD0373
SWD0374
SWD0375
SWD0376
SWD0377
SWD0378
SWD0379
SWD0380
SWD0381
SWD0382
SWD0383
SWD0384
SWD0385

TECHNICAL DISCUSSION

1.0 THE MAIN PROGRAM

1.1 Terminology and Geometric Parameters

Following are definitions of terms used in this technical discussion. The terms are also illustrated in Figure 1.

- base - the larger end of the bay, frustum or fairing
- bay - the conical frustum between two rings plus the ring at the upper end of this conical frustum
- frustum - the conical frustum of fairing consisting of all bays having the same half angle
- nose cap - the spherical segment which closes the top of the fairing.

The external geometry of the fairing is specified by the base diameter of the fairing (DBAS), by the half-angle of each frustum (THTA (NF)), and either the ratio of the top diameter of each frustum to the base diameter of the fairing (DOVDB (NF)), or (mandatory when the frustum is a cylinder) the length of the frustum (ALF (NF)). Frustums are numbered by index NF starting at the base of the fairing. Frustum geometry is completely described by the same parameters used to describe fairing geometry. Bay geometry is described by the base diameter of the bay (DSUBB), the half angle of the bay (THETA), and the length of the bay measured along the axis of symmetry (ALB). Bays within a frustum are numbered by index I starting at the base of the frustum. The outside diameter of a ring associated with a bay is equal to the top diameter of the bay. All dimensions of the ring cross-section are expressed in terms of the material thickness used to form the ring (see Figure 3). This thickness will be one of the standard gauges stored in the program.

1.2 Design Logic

The major logical steps followed in designing a nose fairing are shown in the condensed flow chart in Figure 2. In Appendix A, more detailed information is provided by a program listing which includes detailed comments describing in words the operations being performed. Definitions of the more commonly used variable names in this listing appear in Appendix B. The discussion which follows is supplemental to the information in the flow chart and program listing, and in general follows the sequence of the listing.

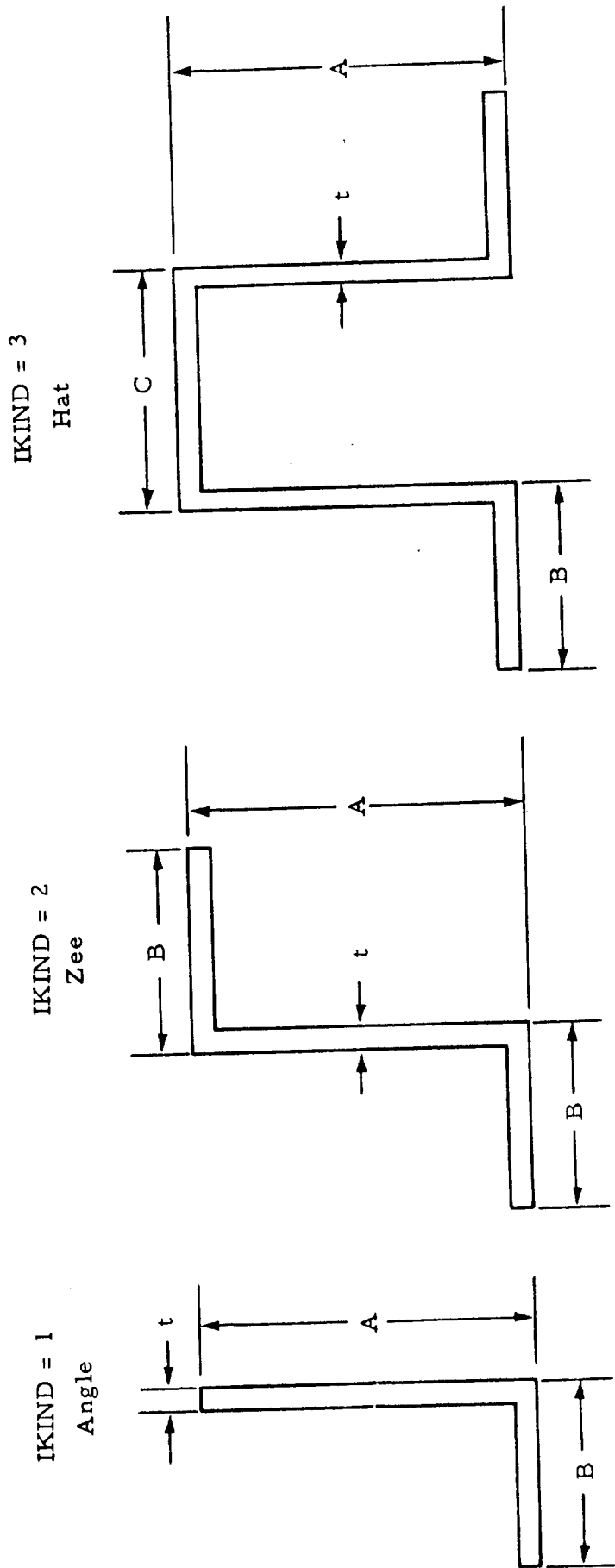


Figure 3 - Stiffener Cross-Section

1.3 Required Input Data

The following input data is required by the computer program:

1. External fairing geometry (Figure 1)
2. Design constraints
 - a. Minimum distance between rings, in.
 - b. Ring type (zee, hat, or angle), also flange and web thickness ratios - see Figure 3.
 - c. Ring outstanding flange buckling stress level, psi.
3. Structural material
 - a. Kind of material to be used. (Certain properties of five materials are stored in the program. See Section 1.5 of Technical Discussion.)
 - b. Additional properties of sandwich face material
 - (1) Tensile allowable stress, psi
 - (2) Compressive allowable stress, psi
 - (3) Shear allowable stress, psi
 - (4) Young's modulus, psi
4. Aerodynamic data at a design point in the trajectory
 - a. Mach number
 - b. Dynamic pressure
 - c. Angle of attack
 - d. Difference between internal pressure of fairing and free-stream pressure
5. Factor of safety (If no value is input, a factor of 1.4 will be used.)
6. Program controls
 - a. Is pressure profile data input? If so, the type of lift data is indicated.
 - b. The desired type of output is indicated. (See Section 1.13.)
7. Pressure profile data (optional). If a pressure profile is not input, it is computed in Subroutine AERO.

Detailed instructions on how to input this data are provided in Section 3.

1.4 The Pressure Profile

Whether input or computed, the system used to specify the pressure profile in the axial direction on the section of fairing composed of conical frustums is illustrated in Figure 4. LT is an index indicating station number, starting with the first station at the junction of the nose cap and top frustum. Uniform spacing between stations is not necessary. Two stations must be located at each intersection of the conical frustums. Where discontinuities in the pressure profile exist, two stations can be indicated for the same location.

The following three parameters are required by the computer program at each station.

1. CPO (LT) - The pressure coefficient at zero angle of attack
2. CPA (LT) - The difference between the pressure coefficient on the windward side when flying at an angle of attack and CPO (LT)
3. XOD (LT) - The axial distance measured from the tip of the nose cap divided by the fairing base diameter

When the pressure profile is input to the program, three options are available for inputting lift data.

1. CPA (LT) as described above
2. CPA (LT) per radian angle of attack
3. $\frac{\partial}{(x/D)} \left(\frac{\partial C_N}{\partial \alpha} \right)$

In Option 3

C_N = the normal force coefficient with the fairing base as a reference area

α = angle of attack in radians

x/D = distance from the leading point in calibers

After they are read into the program, the lift data in Options 2 and 3 are converted to the form of Option 1. A sinusoidal pressure distribution (see Figure 5) in the circumferential direction is used in converting Option 3 to Option 1. Provisions can be made to read in other types of lift parameters, if the parameter can be converted to CPA (LT) after it is input.

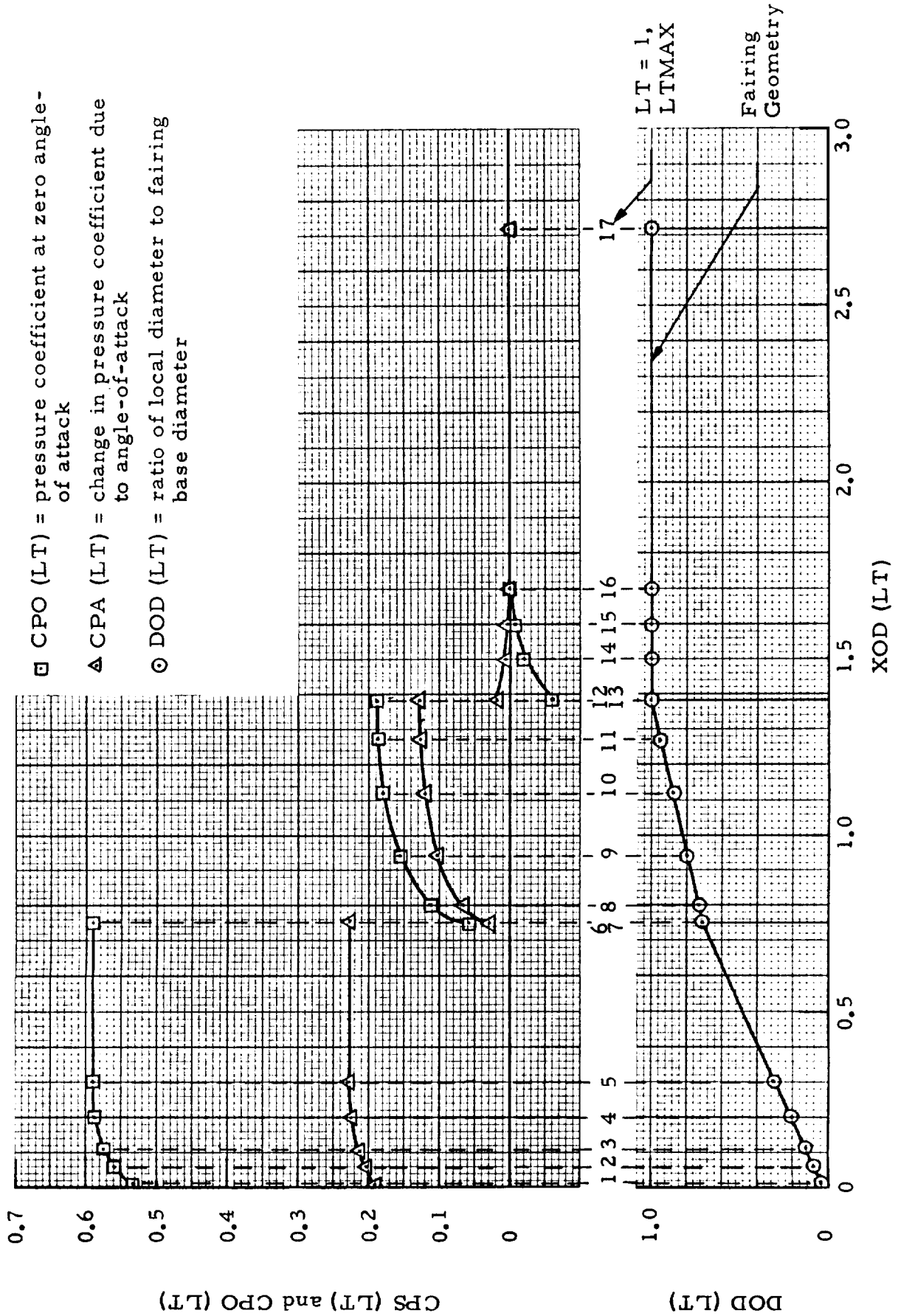


Figure 4 - Pressure Profile Used in Sample Problem

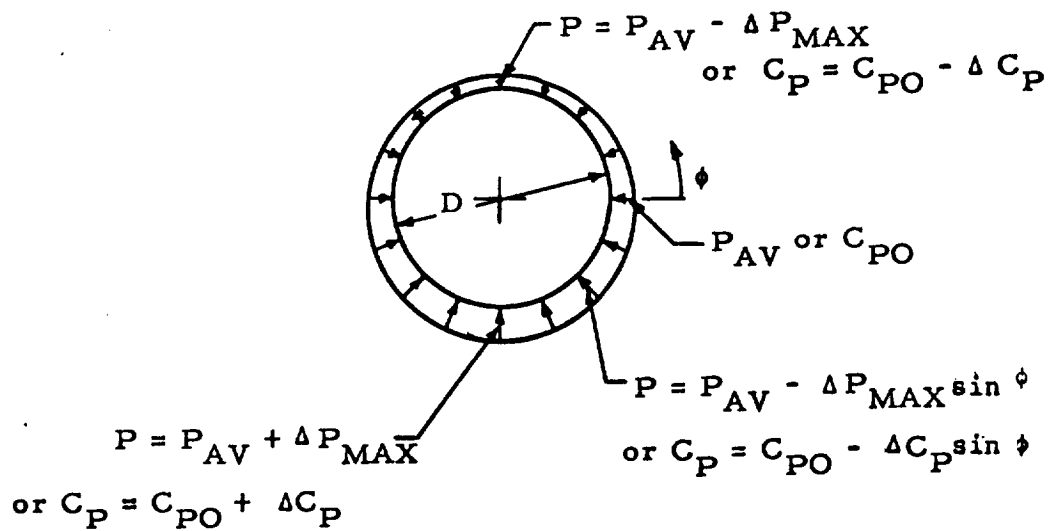


Figure 5 - Circumferential Pressure Distribution

The option is available to either compute or input axial force and lift data for the nose cap. In either case they are specified by the following three parameters:

1. Drag coefficient with the base area of the nose cap as the reference area
2. Normal force coefficient per radian angle-of-attack with the base area of the nose cap as a reference area
3. The location of the center of lifting pressure measured from the base of the nose cap

1.5 Material Properties

Properties for the following five materials are now stored in the program in Subroutine PROPTY:

1. Aluminum
2. Magnesium
3. Titanium
4. Stainless steel
5. Lockalloy

Additional materials can be readily added to this list. Properties which are stored are as follows:

1. Modulus of elasticity
2. Poisson's ratio
3. Density
4. Maximum allowable temperature

If a value is input for maximum allowable temperature the stored value is not used.

The quantities stored in the program are sufficient for most of the program's operations. However, additional data is required for the shell buckling analysis. Since the shell buckling analysis is based upon elastic behavior, it is necessary to input a face cut-off stress level which will ensure that plasticity effects do not become significant.

1.6 Thermal Considerations

The maximum temperature reached by the skin due to aerodynamic heating can be controlled by placing a lower limit on the skin thickness to be used in the nose cap and the top frustum. These minimum thicknesses are computed in Subroutine THERML. The minimum thickness computed for the nose cap is based on heating at the stagnation point, and the minimum

thickness for the top frustum is based on heating at the point at which the nose cap is tangent to the top frustum. The equations in Subroutine THERML are based on a nominal trajectory of two-stage Saturn V vehicle ascending to a 100 nautical mile circular orbit.

1.7 Standard Skin Gauges

Only standard gauges are used in designing stiffening rings. Either standard or non-standard gauges may be used for the sandwich faces and ribbons.

1.8 Design Loads

The individual bay is subjected to bending moments, axial forces and lateral pressure loads. Bending moments and axial forces at the base of the bay are computed in Subroutine LOAD, using the pressure profile data. These loads are then converted to line loads (force per unit of length on the circumference) on both the windward and leeward sides of the bay, and the factor of safety is applied.

The lateral pressure used in design of the bay is the difference between external and internal pressure multiplied by the factor of safety.

1.9 Last Bay in the Frustum

Before beginning the design of a bay, a check is made to determine if there is sufficient length remaining on the frustum for one more bay of minimum length. If there is not sufficient length for one more bay, the length of the last bay designed is added to the remaining undesigned length.

Additional weight is added to the top ring of each frustum to provide for attachment to the next frustum or nose cap.

1.10 Nose Cap Design

Both structural and thermal requirements are considered in the design of the nose cap. The thickness required to limit the temperature to the specified maximum is computed in Subroutine THERML. (For details of the thermal analysis see Appendix J.) The thickness required to withstand aerodynamic loads is computed in Subroutine TNOSST. (For details see Appendix H.) In both cases thickness is determined for conditions at the stagnation point. The greater of these two thicknesses is then used to design an unstiffened cap with uniform skin thickness. Nose cap skin thickness is not limited to standard gauges.

1.11 Output Data

Three options are available on the amount of detail provided by the output data.

1. Design summary only (Figure 6A)

2. Design summary plus design details (Figures 6A, 6B and 6C)
3. Design summary plus design details plus loads details (Figures 6A, 6B, 6C, 6D and 6E).

Most of the headings appearing in the output are self-explanatory. However, there are two which require some comment.

Weight Index - Weight of the bay divided by the inclosed volume of the bay.

Line Load - Force per running inch of circumferences, parallel to the skin surface, normal to the circumferential direction. A factor of safety has been applied to this force.

DATA REPORT FOR OPTIMIZED NOSE FAIRING STRUCTURE
CASE NUMBER 1

MATERIAL = ALUMINUM

AERODYNAMIC LOADS

DYNAMIC PRESSURE, LBS./SQ. FT. QBAR = 765.00
 MACH NUMBER AT DESIGN DYNAMIC PRESSURE AMACH = 1.500
 ANGLE OF ATTACK AT DESIGN DYNAMIC PRESSURE, DEGREES ALPHA = 4.50
 INTERNAL-TU-AMBIENT PRESSURE DIFFERENCE AT DESIGN CONDITIONS, PSI DELTAP = -0.000
 AMBIENT PRESSURE AT DESIGN CONDITIONS, PSI PSTAT = 3.373

CONSTRAINTS ON DESIGN OF FRUSTUMS

MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. TMPMAX = 10000.0
 CRITICAL SECTION SKIN THICKNESS REQUIRED FOR THERMAL REASONS, IN. TCBNTH = 0.0000

DESIGN SUMMARY FOR FRUSTUM SECTION

FRUS- TUM NO.	LARGE DIA. (IN)	SMALL DIA. (IN)	HALF ANGLE (DEG)	LENGTH (IN)	MIN. BAY LENGTH (IN)	MIN. GAUGE (IN)	NO. OF BAYS	USEFUL VOLUME (CU FT)	WEIGHT (LB)
1	260.00	260.00	0.00	549.0	16.6	0.0160	21	9712.96	4194.14
2	260.00	186.91	12.50	164.8	16.4	0.0160	10	3446.23	1148.81
3	186.91	7.80	25.00	192.1	16.6	0.0160	11	940.30	492.18
TOTALS							42	14099.49	5835.12

NOSE CAP DESIGN

DESIGN PRESSURE ON NOSE CAP, PSI PDSPH = 11.396
 MINIMUM ALLOWABLE THICKNESS OF NOSE CAP SKIN, IN. THINN = -0.0000
 MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. TMPMAX = 10000.0
 NOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL INTEGRITY, IN. TCAPST = 0.009
 NOSE CAP SKIN THICKNESS REQUIRED FOR THERMAL REASONS, IN. TCAPM = 0.000
 NOSE CAP SKIN THICKNESS USED TO CALCULATE WEIGHT, IN. TCAP = 0.009
 NOSE CAP RADIUS, IN. RCAP = 4.303
 LENGTH OF NOSE CAP, IN. ALCAP = 1.43
 NOSE CAP SURFACE AREA, SQ. IN. SCAP = 67.18
 USEFUL VOLUME OF NOSE CAP, CU. FT. WCAP = 0.01
 WEIGHT OF NOSE CAP, LBS. WCAP = 0.06

TOTAL LENGTH, VOLUME AND WEIGHT OF FAIRING

TOTAL LENGTH OF FAIRING, IN. ALTOT = 700.37
 USEFUL VOLUME OF FAIRING, CU. FEET VTOT = 14262.64
 TOTAL VOLUME OF FAIRING, CU. FEET VGRSS = 15558.15
 TOTAL WEIGHT OF FAIRING, LBS. WTOT = 5835.18

Figure 6A - Computer Output for Sample Problem - Design Summary

DESIGN DETAILS OF CYLICAL FRUSTUMS

RING DATA

TYPE
BASE LEG H/T
UPRIGHT LEG H/T
OUTSTANDING LEG H/T
FLANGE RUCKLING STRESS, PSI

ZEC SECTION
HWT = 10.00
AWT = 20.00
CWT = 6.00
FCFB = 30000.

SANDWICH FACE DATA

FACE MATERIAL

7075-T6 ALUMINUM

Figure 6B - Computer Output for Sample Problem - Design Details

FRUSTUM* RAY ID.	SHELL NO. (IN)	RAY LENGTH (IN)	FACE GAUGE (IN)	REPRD. GAUGE (IN)	RING GAUGE (IN)	TOTAL FACE WT. (LB)	TOTAL CAKE WT. (LB)	RING WT. (LB)	WEIGHT INDEX (LB/CU FT)	SHELL THICKNESS (IN)	TOTAL RAY WT. (LB)	CELL WEIGHT (IN)
1-1	260.0	16.6	0.0320	0.00100	0.1900	89.47	3.14	109.65	0.4970	0.9070	234.1	1.0000
1-2	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3810	1.1450	197.3	0.5625
1-3	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3810	1.1450	197.3	0.5625
1-4	260.0	16.6	0.0250	0.00100	0.1900	57.57	3.14	109.56	0.4119	0.9000	210.1	0.8125
1-5	260.0	16.6	0.0200	0.00100	0.1900	53.98	4.48	109.34	0.3824	1.2700	193.0	0.5625
1-6	260.0	16.6	0.0200	0.00100	0.1900	53.98	5.30	109.12	0.3851	1.5200	195.4	0.5625
1-7	260.0	16.6	0.0200	0.00100	0.1900	53.95	4.92	109.23	0.3837	1.3950	195.7	0.5625
1-8	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3810	1.1450	197.3	0.5625
1-9	260.0	16.6	0.0200	0.00100	0.1900	54.00	3.59	109.55	0.3796	1.0200	193.0	0.6250
1-10	260.0	16.6	0.0400	0.00100	0.1900	108.11	3.14	109.65	0.3129	0.9150	261.6	1.0000
1-11	260.0	16.6	0.0250	0.00100	0.1900	67.47	4.92	109.23	0.4164	1.4000	213.2	0.8750
1-12	260.0	16.6	0.0200	0.00100	0.1900	53.92	5.36	109.12	0.3851	1.5200	195.4	0.6250
1-13	260.0	16.6	0.0250	0.00100	0.1900	67.53	3.59	109.55	0.3134	0.7700	192.2	0.9375
1-14	260.0	16.6	0.0200	0.00100	0.1900	54.08	2.70	109.77	0.3769	0.7700	192.2	0.6875
1-15	260.0	16.6	0.0200	0.00100	0.1900	53.98	4.48	109.34	0.3824	1.2700	193.0	0.6875
1-16	260.0	16.6	0.0320	0.00100	0.1900	86.61	1.80	109.93	0.4535	0.5320	231.3	1.0000
1-17	260.0	16.6	0.0200	0.00100	0.1900	54.11	2.25	109.87	0.3755	0.6450	191.5	0.7500
1-18	260.0	16.6	0.0200	0.00100	0.1900	54.00	4.03	109.44	0.3810	1.1450	197.3	0.7500
1-19	260.0	16.6	0.0320	0.00100	0.1900	86.36	4.48	109.34	0.4644	1.2120	236.9	1.0000
1-20	260.0	16.6	0.0200	0.00100	0.1250	54.05	3.14	109.44	0.3568	0.8950	131.0	0.8125
1-21	260.0	17.0	0.0200	0.00100	0.1250	55.38	2.76	97.50	0.2524	0.7700	131.9	0.8125
2-1	252.7	16.4	0.0250	0.00100	0.1250	57.24	5.35	44.09	0.3066	1.5250	150.2	1.0000
2-2	245.5	16.4	0.0200	0.00100	0.1250	54.26	5.19	44.75	0.2741	1.5200	126.9	0.8125
2-3	238.2	16.4	0.0200	0.00100	0.1250	50.94	1.69	47.75	0.2741	0.5200	119.5	0.8750
2-4	230.9	16.4	0.0200	0.00100	0.1250	49.11	8.09	41.86	0.3005	1.8950	123.2	0.8750
2-5	223.0	16.4	0.0250	0.00100	0.1250	59.77	2.58	40.95	0.3337	0.7750	125.5	1.0000
2-6	216.4	16.4	0.0200	0.00100	0.1250	46.51	1.92	39.64	0.3026	0.6450	109.2	0.8750
2-7	209.1	16.4	0.0200	0.00100	0.1250	44.35	1.12	38.38	0.3092	0.3950	104.3	0.9375
2-8	201.6	16.4	0.0200	0.00100	0.1250	43.24	1.80	36.93	0.3236	0.6450	101.3	0.9375
2-9	194.6	16.4	0.0200	0.00100	0.1000	41.70	1.73	28.83	0.2918	0.6450	92.4	0.9375
2-10	171.4	17.2	0.0250	0.00100	0.1250	52.71	1.75	42.63	0.3428	0.6500	97.7	1.0000
3-1	156.0	16.6	0.0200	0.00100	0.1250	41.08	1.71	31.21	0.3788	0.6450	91.8	0.6875
3-2	156.0	16.6	0.0200	0.00100	0.1000	37.17	4.69	17.63	0.3882	2.0200	78.5	0.7500
3-3	140.5	16.6	0.0200	0.00100	0.0900	33.92	1.69	16.31	0.4025	0.7700	66.8	0.8125
3-4	125.0	16.6	0.0200	0.00100	0.0900	30.42	1.01	11.79	0.4217	0.5200	56.1	0.8125
3-5	109.5	16.6	0.0200	0.00100	0.0900	26.83	1.11	10.25	0.4783	0.6450	49.7	0.8750
3-6	94.0	16.6	0.0200	0.00100	0.0800	23.25	0.96	6.94	0.5246	0.6450	41.1	0.9375
3-7	78.5	16.6	0.0200	0.00100	0.0710	19.67	0.97	4.53	0.5964	0.7700	33.6	1.0000
3-8	63.1	16.6	0.0200	0.00100	0.0630	16.07	0.92	2.63	0.7035	0.8950	26.7	1.0000
3-9	47.0	16.6	0.0200	0.00100	0.0630	12.46	0.92	2.05	0.9899	1.0200	20.7	1.0000
3-10	32.1	16.6	0.0200	0.00100	0.0500	9.11	0.15	0.91	1.1251	0.2700	13.6	1.0000
3-11	7.8	25.1	0.0320	0.00100	0.0250	11.21	0.23	0.09	2.3595	0.5320	13.6	1.0000

Figure 6C - Computer Output for Sample Problem - Design Details (Cont. 'd)

FRUSTUM -DAY NO.	MAX APPLIED FACE STRESS, PSI		ALLOWABLE STRESS, PSI		
	AXIAL	SHEAR	BURST	FACE MARKING YIELD	FACE DIMPLING
1-1	17534.	3725.	11602.	46058.	22113.
1-2	27274.	4362.	18564.	60048.	27200.
1-3	28494.	4362.	18564.	60048.	27200.
1-4	28571.	3490.	14851.	60048.	20442.
1-5	28534.	4362.	18564.	60048.	27200.
1-6	24155.	4362.	18564.	60048.	27200.
1-7	23175.	4362.	18564.	60048.	27200.
1-8	22395.	4362.	18564.	60048.	27200.
1-9	21915.	4362.	18564.	60048.	27200.
1-10	16512.	2181.	9232.	54192.	34751.
1-11	18204.	3490.	14851.	60048.	17624.
1-12	15476.	4362.	18564.	60048.	22113.
1-13	14957.	3490.	14851.	60048.	15336.
1-14	17516.	4362.	18564.	60048.	22113.
1-15	17136.	4352.	18564.	60048.	18075.
1-16	16223.	2725.	11602.	61222.	22113.
1-17	15577.	4362.	18564.	60048.	15336.
1-18	14797.	4362.	18564.	60048.	15336.
1-19	8761.	2724.	11602.	60048.	22113.
1-20	13238.	4352.	18564.	60048.	13085.
1-21	12662.	4335.	18564.	60048.	13085.
2-1	9564.	3439.	14962.	60048.	13497.
2-2	11582.	4135.	18170.	60048.	13085.
2-3	11211.	3971.	17038.	48955.	1282.
2-4	10344.	3810.	17107.	60048.	11282.
2-5	8345.	2927.	13260.	60048.	13487.
2-6	10121.	3514.	16043.	60048.	11282.
2-7	39053.	3377.	15511.	56170.	9328.
2-8	9405.	3278.	14979.	60048.	9825.
2-9	9046.	2134.	14448.	60048.	9825.
2-10	6550.	2441.	11120.	48955.	13497.
3-1	8950.	3034.	14008.	60048.	18275.
3-2	9196.	2778.	12788.	60048.	15356.
3-3	7441.	2522.	11569.	60048.	13085.
3-4	6884.	2254.	10349.	48955.	13085.
3-5	5927.	2000.	9129.	60048.	11282.
3-6	5169.	1747.	7910.	60048.	9825.
3-7	4410.	1476.	6630.	60048.	4538.
3-8	3632.	1222.	5473.	60048.	8638.
3-9	2803.	960.	4251.	60048.	6038.
3-10	2170.	707.	3031.	67339.	5536.
3-11	512.	269.	877.	61222.	22113.

Figure 6D - Computer Output for Sample Problem - Loads and Stresses

DETAILED LOADS DATA

POSITION NO.	RAY NO.	RAY DIA. (IN)	RAY TYP. (IN)	RAY LENGTH (IN)	DISTANCE FROM BASE (IN)	DES. PRES. WIDTH (KSI)	DES. PRES. LENGTH (FT)	AXIAL LOAD (LBS)	SMALL LAC (LBS)	SCUDING MOMENT (IN-LBS)
1	200.0	260.0	260.0	16.6	0.0	-0.00	-0.00	108426.4	71251.7	3300090.0
1-1	260.0	260.0	260.0	16.6	16.6	-0.00	-0.00	108426.4	71251.7	5612515.1
1-2	260.0	260.0	260.0	16.6	33.2	-0.00	-0.00	108426.4	71251.7	3314200.8
1-3	260.0	260.0	260.0	16.6	49.8	-0.00	-0.00	108426.4	71251.7	3199292.3
1-4	260.0	260.0	260.0	16.6	66.4	-0.00	-0.00	108426.4	71251.7	3077018.3
1-5	260.0	260.0	260.0	16.6	83.0	-0.00	-0.00	108426.4	71251.7	2959337.4
1-6	260.0	260.0	260.0	16.6	99.6	-0.00	-0.00	108426.4	71251.7	2841040.3
1-7	260.0	260.0	260.0	16.6	116.2	-0.00	-0.00	108426.4	71251.7	2727435.2
1-8	260.0	260.0	260.0	16.6	132.8	-0.00	-0.00	108426.4	71251.7	2604434.0
1-9	260.0	260.0	260.0	16.6	149.4	-0.00	-0.00	108426.4	71251.7	24861397.0
1-10	260.0	260.0	260.0	16.6	166.0	-0.00	-0.00	108426.4	71251.7	2367863.3
1-11	260.0	260.0	260.0	16.6	182.6	-0.00	-0.00	108426.4	71251.7	22495708.0
1-12	260.0	260.0	260.0	16.6	199.2	-0.00	-0.00	108426.4	71251.7	21312764.3
1-13	260.0	260.0	260.0	16.6	215.8	-0.00	-0.00	108426.4	71251.7	20129820.5
1-14	260.0	260.0	260.0	16.6	232.4	-0.00	-0.00	108426.4	71251.7	18946876.0
1-15	260.0	260.0	260.0	16.6	249.0	-0.00	-0.00	108426.4	71251.7	17763931.8
1-16	260.0	260.0	260.0	16.6	265.6	-0.00	-0.00	108426.4	71251.7	16580987.6
1-17	260.0	260.0	260.0	16.6	282.2	-0.00	-0.00	108426.4	71251.7	15398043.3
1-18	260.0	260.0	260.0	16.6	298.8	-0.00	-0.00	108426.4	71251.7	14215098.9
1-19	260.0	260.0	260.0	16.6	315.4	-0.00	-0.00	108426.4	71251.7	13034303.1
1-20	260.0	260.0	260.0	17.0	332.0	-0.00	-0.00	108426.4	70817.9	11856044.4
1-21	260.0	260.0	260.0	16.4	348.6	-0.00	-0.00	108426.4	70817.9	10673799.7
2-1	260.0	260.0	260.0	16.4	365.2	-0.00	-0.00	108426.4	65672.7	9501899.6
2-2	260.0	260.0	260.0	16.4	381.8	-0.00	-0.00	108426.4	61241.3	8342414.8
2-3	260.0	260.0	260.0	16.4	398.4	-0.00	-0.00	108426.4	57021.5	7182913.9
2-4	260.0	260.0	260.0	16.4	415.0	-0.00	-0.00	108426.4	53077.2	6023818.1
2-5	260.0	260.0	260.0	16.4	431.6	-0.00	-0.00	108426.4	49379.4	48790284.4
2-6	260.0	260.0	260.0	16.4	448.2	-0.00	-0.00	108426.4	45907.7	3730211.8
2-7	260.0	260.0	260.0	16.4	464.8	-0.00	-0.00	108426.4	42652.8	2583307.9
2-8	260.0	260.0	260.0	16.4	481.4	-0.00	-0.00	108426.4	39744.6	14368312.9
2-9	260.0	260.0	260.0	16.4	498.0	-0.00	-0.00	108426.4	37305.9	2977122.8
3-1	260.0	260.0	260.0	16.6	514.6	-0.00	-0.00	108426.4	35382.2	2350960.1
3-2	260.0	260.0	260.0	16.6	531.2	-0.00	-0.00	108426.4	29233.7	1307528.5
3-3	260.0	260.0	260.0	16.6	547.8	-0.00	-0.00	108426.4	24703.3	1344763.4
3-4	260.0	260.0	260.0	16.6	564.4	-0.00	-0.00	108426.4	19983.5	954462.7
3-5	260.0	260.0	260.0	16.6	581.0	-0.00	-0.00	108426.4	15177.4	687442.7
3-6	260.0	260.0	260.0	16.6	597.6	-0.00	-0.00	108426.4	12031.0	422607.3
3-7	260.0	260.0	260.0	16.6	614.2	-0.00	-0.00	108426.4	8774.5	236638.7
3-8	260.0	260.0	260.0	16.6	630.8	-0.00	-0.00	108426.4	6030.4	104339.0
3-9	260.0	260.0	260.0	16.6	647.4	-0.00	-0.00	108426.4	3702.5	33546.4
3-10	260.0	260.0	260.0	16.6	664.0	-0.00	-0.00	108426.4	2112.5	3308.2
3-11	260.0	260.0	260.0	16.6	680.6	-0.00	-0.00	108426.4	933.9	10579.3
3-12	260.0	260.0	260.0	16.6	697.2	-0.00	-0.00	108426.4	50.7	158.2
3-13	260.0	260.0	260.0	16.6	713.8	-0.00	-0.00	108426.4	229.2	158.2

TOTALS
7.0

Figure 6E - Computer Output for Sample Problem - Detailed Loads

2.0 DESCRIPTION OF SUBROUTINES

2.1 Subroutine THERML (Thermal Computations)

This subroutine computes the minimum thickness of skin required to limit the skin temperature to the specified maximum. Minimum thicknesses are computed for both the nose cap and top frustum. The analytical basis for the computations performed in Subroutine THERML is presented in Appendix J. The analysis is based on a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile circular orbit.

The equations and stored coefficients in Subroutine THERML were developed by means of a multiple regression analysis of a large amount of analytical data generated for the trajectory mentioned above. Details of the multiple regression analysis are also presented in Appendix J.

The parameters required from the main program for computations in Subroutine THERML are RCAP, THETA, TMPMAX and MAT. Returned to the main program are TCONTH and TCAPTH.

2.2 Subroutine TNO SST (Nose Cap Structure)

Subroutine TNO SST calculates the nose cap skin thickness required to withstand the pressure differential, PDSPH, at the stagnation point. The method used to calculate PDSPH and the structural analysis of a spherical nose cap are described in Appendix H.

Parameters required by Subroutine TNO SST are PDSPH, E, RCAP and TMINN. Returned to the main program is TCAPST.

2.3 Subroutine AERO (Pressure Coefficients)

When a pressure profile is not input to the program the pressure coefficient data for the profile is computed in Subroutine AERO. The analytical basis for the computations performed in this subroutine is presented in Appendix C. Because of the assumptions made in computing this data, the pressure coefficient at zero angle-of-attack and the change in pressure coefficient due to angle-of-attack are uniform over the length of each frustum. The pressure profile is then constructed by assigning the pressure coefficient data for each frustum to the beginning and end points of each frustum. Double points occur at the intersection of two frustums.

Parameters required to make the computations in Subroutine AERO are NF, THETA (NF), AMACH and ALPHA. Returned to the main program are CPOO and CPAA.

A sinusoidal pressure distribution in the circumferential direction is used in computing CPAA.

2.4 Subroutine LOAD (Bending Moment, Axial Force and Shear Force)

Using the pressure profile data, Subroutine LOAD computes the bending moment, axial force and shear force at the base of each bay. Bending moment and axial force are then converted to line loads (force per unit length of circumference) for both the windward and leeward sides of the bay, and the factor of safety is applied to these line loads.

When Subroutine LOAD is called upon for loads on the first bay, the bending moment, axial force and shear force are computed at the base of the fairing. As design of the fairing moves from the base to the nose cap, increments of the loads contributed by the pressure profile between the previous bay location and the new bay location are subtracted from the previous totals. Derivation of the equations used in this subroutine appear in Appendix D.

Information required for the computations in Subroutine LOAD is

1. Pressure profile data:

LTMAX

CPO (LT) LT = 1, LTMAX

CPA (LT) LT = 1, LTMAX

XOD (LT) LT = 1, LTMAX

2. Fairing geometry:

NFMAX

DBAS

ALF (NF) NF = 1, NFMAX

THTA (NF) NF = 1, NFMAX

3. Aerodynamic data:

AMACH

QBAR

DELTAP

ALPHA

4. Miscellaneous:

DSUBB

C2

FS

LFLG

LTRIG

Parameters computed by Subroutine LOAD are ANFIMX, ANFIMN, FSUBZ and LTUNCT (NF) for NF = 1 to NFMAX.

2.5 Subroutine PRESUR (Lateral Pressure)

Using pressure profile data, Subroutine PRESUR computes the pressure differential across the skin along the length of the bay. The maximum differential occurring along the bay length is determined for both the windward and leeward sides, multiplied by the factor of safety and returned to the main program. The equations used in this subroutine are developed in Appendix E.

Parameters required for computations performed in Subroutine PRESUR are

1. Pressure profile data

LTMAX

CPO (LT) LT = 1, LTMAX

CPA (LT) LT = 1, LTMAX

XOD (LT) LT = 1, LTMAX

2. Other parameters: LPFL, C2, DBAS, DSUBB, ALB, QBAR, NFMAX, NF, DELTAP, FS and LTUNCT (NF) for NF = 1 to NFMAX.

Parameters computed in Subroutine PRESUR are PDESMN, PDESMX, ALN and ALX

2.6 Subroutine PROPTY (Material Properties)

Properties of several commonly used materials are stored in this subroutine.

The material for which properties are desired is indicated by the input parameter MAT.

Output parameters are the material properties E, AMU, RHO, and TMPMAX.

2.7 Subroutine DIAM (Local Diameter)

When given the parameters describing the external geometry of the fairing and the distance from the tip of the nose cap in calibers this subroutine computes the local fairing diameter.

Parameters required by this subroutine are:

ALTOT

ALF (NF), NF = 1, NFMAX

THTA (NF), NF = 1, NFMAX

DMN (NF), NF = 1, NFMAX

XN

NFMAX

The local diameter, DLOC, is computed by this subroutine.

2.8 Subroutine DLOD (Incremental Loads)

When given the geometry and pressure coefficient data for an incremental length of the fairing this subroutine computes the contribution by this increment to the total bending moment, axial load and shear load.

Input parameters are XOD1, XOD2, D1, D2, CP01, CP02, CPA1, CPA2, A3, A4, and DP.

Parameters computed in Subroutine DLOD are FSBZ, BND, and AXLOD.

2.9 Subroutine RING (Ring Strength and Stiffness)

When given the cross-sectional shape of the ring and the skin gauge from which it is to be fabricated, this subroutine computes the moment of inertia of the ring with and without the effective skin, its cross-sectional area, eccentricity and torsion constant. This subroutine also calls Subroutine RSTRES which computes the local stress level in the ring.

Input parameters required by RING are E, IKIND, AOT, BOT, COT, WSEF, J, K, PI, and T(J) when J = 1, 30.

Output parameters are AIRING, AISTT, AST, A, Z, ECC and TCONST.

2.10 Subroutine RSTRES (Local Stress Level in Ring)

When designing a stiffener ring it is necessary to check for local instability in the ring flange or web (see Appendix G). Subroutine RSTRES computes the local stress level in the ring. This computed stress level is then compared to an input flange buckling stress level, and, if necessary, a greater web thickness is assigned to the ring.

Input parameters for Subroutine RSTRES are A, T(J), Z, PDESMX, DSUBB, CTH, AL(J), AST, and AISTT.

The output parameter is FRING.

2.11 Subroutine IREQ (Required Moment of Inertia)

This subroutine computes the stiffening ring moment of inertia requirements to prevent general instability of the structure. The methods used to compute this required moment of inertia are described in Appendix G.

Input parameters are THTA (NF), C6, AL(J), DSUBB, C2, PDESMX, T(J), PI, ANFIMX, E and ALCONE.

The output parameter is AIREQ.

2.12 Subroutine INSTBL (Panel Stability)

When given details of a tentative sandwich design for a bay, along with applied crushing or bursting pressure, INSTBL computes the line load at which the bay will fail due to overall instability. The analysis was prepared by B. O. Almroth of LMSC's Solid Mechanics Laboratory, and is described in Appendix K.

Subroutine INSTBL requires a large amount of tabular information which, for reasons of convenience, is read into the program as regular input data. This data is physically located immediately after the program deck. Nose fairing data cards for a number of nose fairing designs can be stacked in the usual manner behind these cards.

Input parameters required by subroutine INSTBL are EFACE, G1, G2, AMU, T1, T2, APRC, H, KY, PXL, and KG. Output from the subroutine is CRG, the initial line load, which is compared to the applied line load at the position (windward or leeward) then under investigation.

3.0 INPUT FORMAT

When the pressure profile and nose cap lift and drag data are to be computed in the program, the following three types of input data cards are the only types required.

Types 1, 2, 3, and 4 Parameters which apply to the entire fairing.

Type 5 Parameters which apply to individual frustum (one card per frustum).

When lift and drag data for the nose cap are to be input, or when a pressure profile is to be input, the following additional card type is required.

Type 6 Lift and drag data for the nose cap.

When pressure profile data is to be input, two additional card types are required.

Type 7 Pressure profile data points (one card per data point.)

Type 8 Card indicating end of pressure profile data.

The detailed format for these cards is as follows:

Type 1: Format (5F12.8, 2I6)

Data:	DBAS	- Base diameter of fairing, in.
	QBAR	- Dynamic pressure at design point in the trajectory, lbs/sq. ft.
	AMACH	- Mach number at design point in the trajectory.
	ALPHA	- Angle-of-attack at design point in the trajectory, degrees.
	KEY	- A integer indicating the type of output desired. The code is as follows:
		(0) Design summary only. (See Figure 6A).
		(1) Design summary plus design details (Figures 6B and 6C.)
		(2) Design summary plus design details plus load details (Figures 6C, 6D and 6E.)

Type 2: Format (2I6, 4F12.8, I12)

Data:	MAT	- An integer indicating the material to be used. The code is as follows:
		(1) aluminum
		(2) magnesium

- (3) titanium
 - (4) stainless steel
 - (5) Lockalloy (a Be-Al alloy)
- NFMAX - The number of conical frustums in the fairing.
- TMP - The maximum allowable skin temperature for the nose cap and top frustum, °F. (If no value is input, the value stored with the material properties is used. If a value equal to or greater than 10,000 is input, no thermal constraint is imposed on the skin thickness.)
- TMINN - The minimum skin gauge to be used in nose cap design, in.
- DELTAP - Difference between internal and free-stream pressure, psi
- FS - Factor of safety. (If no value is input, a factor of safety of 1.4 is used.)
- LPRES - An integer indicating whether nose cap lift and drag data and/or pressure profile data will be input. If pressure profile data is input, LPRES also indicates the type of lift data to be input. The code is as follows:
- (-1) Lift and drag data for the nose cap is input, but no pressure profile data is input.
 - (0) No nose cap or pressure profile data is input.
 - (1) Nose cap data and pressure profile data with CPA (LT) as defined in Figure 5 and Appendix B is input on Card type 5.
 - (2) Same as (1) except that CPA (LT) per radian angle-of-attack is input for CPA (LT) on Card type 5.
 - (3) Same as (1) except that

$$\frac{\partial}{\partial X} \left(\frac{\partial C_N}{\partial \alpha} \right) \text{ (See Section 1.4) is input for CPA (LT) on Card type 5.}$$

Type 3: Stringer Data (I5, 6E12.8)

Data: **IKIND** - Type of cross-section

 1 = angle

 2 = zee

 3 = hat

AOT - Web height-to-thickness ratio (see Figure 3)

BOT - Flange width-to-thickness ratio (see Figure 3)

COT - Hat section flange width-to-thickness ratio
(see Figure 3)

FCFB - Ring outstanding flange buckling level, psi

Type 4: Face Material (2I6, 4F12.8, I12)

Data: **MATF** - Type of material

 1 = 2024-T4 Alclad Aluminum

 2 = 2024-T4 Aluminum

 3 = 7075-T6 Aluminum

FTA - Face tensile allowable stress

FCA - Face compressive allowable stress

FSA - Face shear allowable stress

EFACE - Modulus of elasticity of face material

One Type 5 card is required for each frustum.

Type 5: (5F12.8, 2I6)

Data: **ALF (NF)** - Ratio of top diameter of frustum to base
diameter of fairing or length of the frustum
in inches. If the number is equal to or
greater than 1, it will be treated as frustum
length. For conical sections, either the
diameter ratio or length can be used. For
cylindrical sections, only length can be input.

THTA (NF) - Frustum half angle, degrees.

- ELMIN (NF) - Bay length to be used in designing frustum, in.
- TMNC (NF) - Minimum face thickness to be used in designing frustum, in.
- TRBMIN (NP) - Minimum ribbon thickness to be used in designing frustum, in.
- KTRSTD - An integer indicating whether standard ribbon gauge material is to be used
 - 0 - Use non-standard gauges
 - 1 - Use standard gauges
- KTFSTD - An integer indicating whether standard gauge material is to be used for sandwich faces.
 - 0 - Use non-standard gauges
 - 1 - Use standard gauges

The next card type is required only when nose cap lift and drag data or a pressure profile is read in (PRES = -1, 1, 2 or 3 in Card Type 2). If a blank card is inserted for Card Type 6 when LPRES = 1, 2 or 3, the program will compute CDCAP, CNCAP and XBCAP.

Type 6: Format (3F12.8)

- Data: CDCAP - Spherical nose cap drag coefficient with nose cap base area as a reference area.
- CNCAP - Normal force coefficient per radian angle-of-attack for the nose. Reference area is nose cap base area.
- XBCAP - Distance from base of nose cap to center of pressure for the nose cap, in.

The next two card types are required only when pressure profile data is input (LPRES = 1, 2 or 3). (See Section 1.4 and Figure 4.)

Type 7: Format (3F12.8)

- Data: CPO (LT) - Zero angle-of-attack pressure coefficient at station LT.
- CPA (LT) - Lift parameter at station LT. See LPRES on Card Type 2 and Section 1.4 for options which are available.
- XOD (LT) - Local to base diameter ratio at station LT.

(One card is required for each data point, starting with the first point at the junction of the nose cap and top frustum.)

Type 8: Format (71X, 11)

Data: LSTOP = 1 - This signals the computer that the last pressure profile data point has been read in.

The set of data cards described above will design one fairing. A number of fairings can be designed with one computer run by placing several sets of data cards behind the program deck.

4.0 SAMPLE PROBLEM

The following sample problem illustrates the input format of the program. Input data is as follows:

Fairing geometry:

DBAS	= 260.0 inches
NFMAX	= 3
ALF (1)	= 349.0 inches
ALF (2)	= 0.72
ALF (3)	= 0.03
THTA (1)	= 0.00 degrees
THTA (2)	= 12.5 degrees
THTA (3)	= 25.0 degrees

Design specifications:

ELMIN (1)	= 16.6 inches
ELMIN (2)	= 16.4 inches
ELMIN (3)	= 16.6 inches
TMNC (1)	= 0.016 inches
TMNC (2)	= 0.016 inches
TMNC (3)	= 0.016 inches
KTFSTD	= 1 (use standard gauges for faces)
TRBMIN (1)	= 0.001 inches
TRBMIN (2)	= 0.001 inches
TRBMIN (3)	= 0.001 inches
KTRSTD	= 1 (use standard gauges for ribbon)
TMINN	= 0.0 inches
TMP	= 10000.0°F

One ring shape is specified for all three frustums.

IKIND	= 2
AOT	= 20.0
BOT	= 10.0
COT	= 0.0
FCFB	= 30000.0 psi

Material:

MAT = 1 (aluminum)

Face Material:

MATF = 1 (7075-T6 Aluminum)
 FTA = 58000. psi
 FCA = 41500. psi
 FSA = 27000. psi
 EFACE = 10300000. psi

Aerodynamic data:

AMACH = 1.5
 QBAR = 765.0 lbs/sq ft
 ALPHA = 8.5 degrees
 DELTAP = 0. psi

Factor of Safety:

FS = 1.4 (It is not necessary to input this value,
 since 1.4 is the value which the program
 uses when no value is indicated.)

Program Controls:

LPRES = 1
 KEY = 2
 DELTAL = 0.1 inches

Nose cap lift and drag data:

A blank card is inserted in the deck in place of Card Type 4,
 causing the program to compute this data.

Pressure profile data:

Data for the pressure profile is taken from Figure 4 and listed
 in Figure 8 under Card Type 7.

This input data is arranged in key-punch format in Figure 7. The
 computer output for this problem is shown in Figures 6A, 6B, 6C and 6D.

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APPENDIX A
MAIN PROGRAM LISTING

```

$JOB          LMSC-C53/ADAMS SAB ,460040,00,12,140CEP
$*           NOSE FAIRING OPTIMIZATION PROGRAM
$EXECUTE     IBJOB
$IBJOB LMSC  MAP
$IBFTC C53S  DECK
C            STRUCTURAL OPTIMIZATION AND DESIGN OF MULTI-FRUSTUM NOSE
C            FAIRINGS USING RING AND SANDWICH CONSTRUCTION.
          DIMENSION WSEG(400), VSEG(400), T(30), D(400), ALOPT(400) ,
          1TRING(400), WRING(400), WIDEX(400), ENFIMX(400), ENFIMN(400),
          2FFWR(400),FFDMP(400),ELMIN(400),TMNC(400),IMX(400),CELSIZ(400),
          3FZ(400),TRBN (400),WFACE(400),WTOTFC(400),TRBMIN(400),TRBSTD( B),
          4 WTCORE(400),TFACE(400),FFAX(400),FFC(400),FFS(400),FFB(400),
          5 WSPICE(400), TCORE(400)
          INTEGER CASE
          COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
          COMMON /THRML/ TMPMAX,MAT,TCONTH,TCAPTH,THETA
          COMMON/AERPRS/CPAA,CPOO,LPFL
          COMMON/INSTB/EFACE, G1, G2,AMU, T1, T2, APRC, H , CRG, KY, P, KG
          COMMON/LOADS/CDCAP,CNCAP,XBCAP,FSUBZ,AXLDCP,FSBZCP,BNDCAP,LTRIG,
          1 ALCAP
          COMMON/KLSS/XL
          COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP, FS,LTMAX,SUMAL,
          1 QBAR,ALTOT
          COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
          1  CHKWND,CHKLEE,C2
          COMMON/SFCTN/ GBAR, F
          COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
          COMMON/AERLOD/ALPHA,AMACH,NF
          COMMON/IRG/ TTM, PI, ALCONE, C6, AIREQ
          COMMON/RNG/ IKIND, AOT, BOT, COT, TSTD(30), K, AIRING, DD
          COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
          COMMON/TNOS/TMINN,PDSPH,TCAPST,RCAP
C
          CASE      = 0
          PI        = 3.1415927
C            READ PARAMETERS WHICH APPLY TO ENTIRE FAIRING.
          10 CONTINUE
          READ (5,131) DBAS,DELTAL,QBAR,AMACH,ALPHA,KEY
          READ (5,132) MAT,NFMAX,TMP,TMINN,DELTAP,FS,LPRES

```

SWD0001
SWD0002
SWD0003
SWD0004
SWD0005
SWD0006
SWD0007
SWD0008
SWD0009
SWD0010
SWD0011
SWD0012
SWD0013
SWD0014
SWD0015
SWD0016
SWD0017
SWD0018
SWD0019
SWD0020
SWD0021
SWD0022
SWD0023
SWD0024
SWD0025
SWD0026
SWD0027
SWD0028
SWD0029
SWD0030
SWD0031
SWD0032
SWD0033
SWD0034
SWD0035
SWD0036
SWD0037
SWD0038
SWD0039

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SWD0040 READ(5,989) IKIND ,AOT,BOT,COT,FCFB
SWD0041 IF (NFMX .GT. 10) GO TO 98
SWD0042     READ SANDWICH FACE MATERIAL DATA. DO NOT USE ALLOWABLE
SWD0043 STRESSES IN EXCESS OF PROPORTIONAL LIMIT, SINCE BUCKLING
SWD0044 ANALYSES ASSUME ELASTIC BEHAVIOR.
SWD0045 READ (5,990) MATF, FTA, FCA, FSA, EFACE
SWD0046 ALTOT = 0.
SWD0047 DMX = DBAS
SWD0048 DO 30 NF = 1,NFMX
SWD0049     READ PARAMETERS WHICH DESCRIBE INDIVIDUAL FRUSTUMS.
SWD0050 READ(5,131) ALF(NF),THTA(NF),ELMIN(NF),TMNC(NF),TRBMIN(NF),KTRSTD,SWD0050
SWD0051 1KTFSTD
SWD0052 THETA = 0.0174532925*THTA(NF)
SWD0053 IF (ALF(NF) .LT. 1.) GO TO 20
SWD0054 DMN(NF) = DMX - 2.*ALF(NF)*SIN(THETA)/COS(THETA)
SWD0055 GO TO 25
SWD0056 20 CONTINUE
SWD0057 DMN(NF) = DBAS*ALF(NF)
SWD0058 ALF(NF) = (DMX-DMN(NF))*COS(THETA)*.5/SIN(THETA)
SWD0059 25 CONTINUE
SWD0060 DMX = DMN(NF)
SWD0061 ALTOT = ALTOT + ALF(NF)
SWD0062 30 CONTINUE
SWD0063 DMX = DBAS
SWD0064     CHECK TO DETERMINE IF PRESSURE PROFILE DATA IS TO BE
SWD0065     READ IN.
SWD0066 IF (LPRES .EQ. 0) GO TO 45
SWD0067     READ NOSE CAP LIFT AND DRAG DATA IF INPUT INSTEAD OF
SWD0068     COMPUTED.
SWD0069 READ (5,131) CDCAP,CNCAP,XBCAP
SWD0070 IF (LPRES .LT. 0) GO TO 50
SWD0071     READ PRESSURE PROFILE DATA.
SWD0072     C
SWD0073     C
SWD0074 DO 35 LT = 1,101
SWD0075 READ (5,365) CPO(LT),CPA(LT),XOD(LT),LSTOP
SWD0076 IF (LSTOP .GT. 0) GO TO 40
SWD0077 35 CONTINUE

```



```

40 CONTINUE
   LTMAX = LT - 1
   GO TO 50
45 CONTINUE
   CDCAP = 0.
50 CONTINUE
   CASE = CASE + 1
      WRITE OUT HEADINGS FOR DATA REPORT.
      WRITE (6,501)
      WRITE (6,550)
      WRITE (6,555) CASE
      IF (FS .LT. 1.) FS = 1.4
         RADIUS OF SPHERICAL NOSE CAP IS COMPUTED.
         THETA = 0.0174532925 * THTA(NFMAX)
         RCAP = 0.5*DMN(NFMAX)/COS(THETA)
         ALCAP = RCAP*(1.-SIN(THETA))
         ALTOT = ALTOT + ALCAP
      IF NOSE CAP RADIUS IS ZERO, A NOTE IS WRITTEN OUT, AND
      MAXIMUM ALLOWABLE TEMPERATURE IS SET EQUAL TO 10000. THIS
      CAUSES THE PROGRAM TO BY-PASS THE THERMAL EQUATIONS, WHICH
      ARE NOT VALID FOR A NOSE CAP RADIUS OF ZERO.
      IF (RCAP .GT. 0.) GO TO 55
      WRITE (6,178)
      TMP = 10000.
55 CONTINUE
      PROPERTIES ARE LOOKED UP FOR THE INPUT MATERIAL. IF
      NO MAXIMUM TEMPERATURE WAS SPECIFIED, A NOMINAL VALUE WILL
      BE CHOSEN AND USED.
      CALL PROPTY (MAT, E, AMU, RHO, TMPMAX )
      CB = SQRT (1. - AMU**2)
      C4 = (PI**2 *E) / (12. * CB)
      IF MAXIMUM ALLOWABLE TEMPERATURE IS EQUAL TO OR GREATER
      THAN 10000., EITHER FROM INPUT DATA OR BECAUSE NOSE CAP
      RADIUS EQUALS ZERO, THE REQUIRED THERMAL THICKNESS OF BOTH
      THE NOSE CAP AND TOP FRUSTUM ARE SET EQUAL TO ZERO.
      IF (TMP .LT. 10000.) GO TO 60
      TMPMAX = TMP

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SWD0078
 SWD0079
 SWD0080
 SWD0081
 SWD0082
 SWD0083
 SWD0084
 SWD0085
 SWD0086
 SWD0087
 SWD0088
 SWD0089
 SWD0090
 SWD0091
 SWD0092
 SWD0093
 SWD0094
 SWD0095
 SWD0096
 SWD0097
 SWD0098
 SWD0099
 SWD0100
 SWD0101
 SWD0102
 SWD0103
 SWD0104
 SWD0105
 SWD0106
 SWD0107
 SWD0108
 SWD0109
 SWD0110
 SWD0111
 SWD0112
 SWD0113
 SWD0114
 SWD0115
 SWD0116

```

C          TCONTH = 0.0
C          TCAPTH = 0.0
C          GO TO 65
C          IF THE INPUT VALUE FOR MAXIMUM ALLOWABLE TEMPERATURE
C          IF EQUAL TO 0., THE STORED VALUE IS USED. IF THE INPUT VALUE
C          IF GREATER THAN 0. BUT LESS THAN 1000., THE INPUT VALUE IS
C          USED FOR MAXIMUM ALLOWABLE TEMPERATURE.
C          60 IF (TMP.GT. 0.) TMPMAX = TMP
C          THE SKIN THICKNESSES REQUIRED TO KEEP SKIN TEMPERATURE
C          BELOW THE MAXIMUM SPECIFIED ARE COMPUTED FOR BOTH THE NOSE
C          CAP (TCAPTH) AND THE TOP FRUSTUM (TCONTH) IN SUBROUTINE
C          THERML.
C          CALL THERML
C          65 CONTINUE
C          DATA T / 0.001, 0.020, 0.040, 0.060, 0.080, 0.100, 0.120, 0.140,
C          1 0.160, 0.180, 0.200, 0.220, 0.240, 0.260, 0.280, 0.300, 0.320,
C          2 0.340, 0.360, 0.380, 0.400, 0.500, 0.600, 0.700, 0.800, 0.900,
C          31.000, 1.250, 1.500, 2.000 /
C          STANDARD RIBBON THICKNESSES
C          DATA TRBSTD / .0007, .001, .0015, .002, .0025, .003, .004, .005 /
C          AMBIENT PRESSURE AT DESIGN CONDITIONS AND DESIGN PRESSURES
C          ON THE NOSE CAP ARE COMPUTED.
C          PSTAT = QBAR / ( 100.8 * AMACH**2)
C          PDSPH = FS*(PSTAT*((166.92158*(AMACH)**7.)/((7.*(AMACH)**2)
C          -1.))**2.5)-PSTAT - DELTAP)
C          1
C          AERODYNAMIC DATA ARE WRITTEN OUT.
C          WRITE (6,307)
C          WRITE (6,72) QBAR
C          WRITE (6,73) AMACH
C          WRITE (6,74) ALPHA
C          WRITE (6,163) DELTAP
C          WRITE (6,162) PSTAT
C
C          DESIGN CONSTRAINTS FOR FRUSTUM SECTION ARE WRITTEN OUT.
C          WRITE (6,308)
C          WRITE (6,96) TMPMAX
C          WRITE (6,164) TCONTH
C

```

SWD0117
 SWD0118
 SWD0119
 SWD0120
 SWD0121
 SWD0122
 SWD0123
 SWD0124
 SWD0125
 SWD0126
 SWD0127
 SWD0128
 SWD0129
 SWD0130
 SWD0131
 SWD0132
 SWD0133
 SWD0134
 SWD0135
 SWD0136
 SWD0137
 SWD0138
 SWD0139
 SWD0140
 SWD0141
 SWD0142
 SWD0143
 SWD0144
 SWD0145
 SWD0146
 SWD0147
 SWD0148
 SWD0149
 SWD0150
 SWD0151
 SWD0152
 SWD0153
 SWD0154

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C           HEADINGS FOR SUMMARY REPORT ARE WRITTEN OUT.
      WRITE (6,309)
      WRITE (6,310)
      WRITE (6,311)
      WRITE (6,312)
      WRITE (6,607)
C           CONVERT ANGLE OF ATTACK FROM DEGREES TO RADIAN.
      ALPHA = 0.0174532925 * ALPHA
      CHECK TO DETERMINE IF PRESSURE PROFILE DATA HAS BEEN
      INPUT.
C           IF (LPRES .GT. 0) GO TO 398
      SINCE PRESSURE DATA WAS NOT INPUT, PRESSURE PROFILE
      DATA IS COMPUTED IN SUBROUTINE AERO.
      ALPR = ALTOT
      LTMAX = 2*NFMAX
      LT = LTMAX + 1
      DO 70 NF = 1, NFMAX
      CALL AERO
      LT = LT - 1
      CPO(LT) = CPOO
      CPA(LT) = CPAA
      XOD(LT) = ALPR/DBAS
      ALPR = ALPR - ALF(NF)
      LT = LT - 1
      CPO(LT) = CPOO
      CPA(LT) = CPAA
      XOD(LT) = ALPR/DBAS
      70 CONTINUE
      GO TO 437
      398 CONTINUE
C           PRESSURE PROFILE DATA WAS INPUT. THE TYPE OF DATA IS
      INDICATED BY LPRES.
      GO TO (437,438,440) ,LPRES
      438 CONTINUE
C           SLOPE OF THE NORMAL FORCE COEFFICIENT CURVE HAS BEEN
      INPUT. THESE VALUES ARE CONVERTED TO THE MAXIMUM CHANGE
      CAUSED IN THE LOCAL PRESSURE COEFFICIENT BY THE SPECIFIED
      ANGLE OF ATTACK.

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SWD0155
 SWD0156
 SWD0157
 SWD0158
 SWD0159
 SWD0160
 SWD0161
 SWD0162
 SWD0163
 SWD0164
 SWD0165
 SWD0166
 SWD0167
 SWD0168
 SWD0169
 SWD0170
 SWD0171
 SWD0172
 SWD0173
 SWD0174
 SWD0175
 SWD0176
 SWD0177
 SWD0178
 SWD0179
 SWD0180
 SWD0181
 SWD0182
 SWD0183
 SWD0184
 SWD0185
 SWD0186
 SWD0187
 SWD0188
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 SWD0190
 SWD0191
 SWD0192

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SWD0193
SWD0194
SWD0195
SWD0196
SWD0197
SWD0198
SWD0199
SWD0200
SWD0201
SWD0202
SWD0203
SWD0204
SWD0205
SWD0206
SWD0207
SWD0208
SWD0209
SWD0210
SWD0211
SWD0212
SWD0213
SWD0214
SWD0215
SWD0216
SWD0217
SWD0218
SWD0219
SWD0220
SWD0221
SWD0222
SWD0223
SWD0224
SWD0225
SWD0226
SWD0227
SWD0228
SWD0229
SWD0230
SWD0231

DO 439 LT = 1,LTMAX
  XN = XOD(LT)*DBAS
  CALL DIAM (XN,DLOC)
  DOD = DLOC/DBAS
  CPA(LT) = 0.5*ALPHA*CPA(LT)/DOD
  GO TO 437
439 CONTINUE
440 CONTINUE
C
C THE CHANGE IN SURFACE PRESSURE COEFFICIENT PER RADIAN
C ANGLE-OF-ATTACK HAS BEEN INPUT. THESE VALUES ARE CONVERTED
C TO THE MAXIMUM CHANGE CAUSED IN THE LOCAL PRESSURE
C COEFFICIENT BY THE SPECIFIED ANGLE-OF-ATTACK.
DO 441 LT = 1,LTMAX
  CPA(LT) = ALPHA*CPA(LT)
441 CONTINUE
437 CONTINUE
C
C AT THIS POINT THE PRESSURE PROFILE DATA AT SPECIFIED
C DIAMETER RATIOS IS IN THE FORM OF SURFACE PRESSURE
C COEFFICIENT AT ZERO ANGLE-OF-ATTACK AND THE MAXIMUM CHANGE
C IN PRESSURE COEFFICIENT DUE TO ANGLE-OF-ATTACK.
C
C PARAMETERS USED IN DESIGN OF THE ENTIRE FAIRING ARE
C INITIALIZED.
SUMAL = 0.
VGROSS = 0.0
I = 0
WTOT = 0.
VTOT = 0.
DBASE = DBAS
LPFL = 1
LTRIG = -1
DO-LOOP WHICH DESIGNS THE FRUSTUMS IS INITIALIZED.
C
C DO 129 NF = 1,NFMAX
C CONSTANTS ARE EVALUATED AND PARAMETERS ARE INITIALIZED
C FOR THE FRUSTUM BEING DESIGNED.
  THETA = 0.0174532925*THTA(NF)
  CTH = COS(THETA)
  STH = SIN(THETA)
  C1 = CB / (0.3 * CTH)
  C2 = 2.*STH/CTH

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SWD0232
 SWD0233
 SWD0234
 SWD0235
 SWD0236
 SWD0237
 SWD0238
 SWD0239
 SWD0240
 SWD0241
 SWD0242
 SWD0243
 SWD0244
 SWD0245
 SWD0246
 SWD0247
 SWD0248
 SWD0249
 SWD0250
 SWD0251
 SWD0252
 SWD0253
 SWD0254
 SWD0255
 SWD0256
 SWD0257
 SWD0258
 SWD0259
 SWD0260
 SWD0261
 SWD0262
 SWD0263
 SWD0264
 SWD0265
 SWD0266
 SWD0267
 SWD0268
 SWD0269

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C3 = 0.3/CTH
C5 = PI / CTH
C6 = (0.25/CTH**2)*(DBASE/(5.51*E*C2))**1.33333
C7 = STH/CTH
DMIN = DMN(NF)
NBAY = 0
ALCONE = ALF(NF)
ALB = ELMIN(NF)
TMINC = TMNC(NF)
WCONE = 0.
SLOPT = 0.
VUSE = 0.

C IDENTIFY CONE BASE DIAMETER AS BASE DIAMETER OF BOTTOM
C SEGMENT.
DSUBB = DBASE
C IF THE TOP FRUSTUM IS BEING DESIGNED, REQUIRED THERMAL
C THICKNESS AND MINIMUM ALLOWABLE THICKNESS ARE COMPARED. THE
C GREATER OF THE TWO IS USED AS MINIMUM ALLOWABLE THICKNESS
C FOR THE TOP FRUSTUM.
IF (NFMAX.GT. NF) GO TO 101
IF (TCONTH.GT. TMINC) TMINC = TCONTH
C DESIGN OF THE INDIVIDUAL BAY BEGINS AT THIS POINT.
101 CONTINUE
J = 2
I = I + 1
IF (I.GT. 400) GO TO 98
C THE REMAINING LENGTH OF THE FRUSTUM IS COMPUTED.
ALMX2 = ALF(NF)-SLOPT
IF((2.*ALB).GT. ALMX2) ALB = ALMX2
C LINE LOADS AND SHEAR LOADS AT THE BASE OF THE BAY ARE
C COMPUTED IN SUBROUTINE LOAD. THEN REIDENTIFIED AND STORED
C FOR OUTPUT PURPOSES.
CALL LOAD
ENFIMN(I) = ANFIMN
ENFIMX(I) = ANFIMX
FZ(I) = FSUBZ
C MAXIMUM DESIGN PRESSURES ALONG THE LENGTH OF THE BAY
C ARE COMPUTED ON BOTH THE WINDWARD AND LEEWARD SIDES.

```

CALL PRESUR = 2
LPFL = 2
MINIMUM MECHANICAL PROPERTIES OF HONEYCOMB MATERIALS ARE LISTED.
 $G = E / (2.0 * (1.0 + AMU))$
 $GFACE = EFACE / (2. * (1. + AMU))$
CORE MATERIAL IS 2024-T4 ALUMINUM
 $CTA = 40000.$
 $CCA = 40000.$
 $CSA = 23000.$
 $ECORE = E$
 $GCORE = E / (2. * (1. + AMU))$
CALCULATE SKIN THICKNESS REQUIRED TO CARRY THE MERIDIANAL COMPRESSIVE LOAD PER INCH
 $TKMCL = ANFIMX / FCA$
CALCULATE SKIN THICKNESS REQUIRED FOR THE CIRCUMFERENTIAL LOAD PER INCH
 $TKCIL = PDESMX * DSUBB / (2.0 * FCA * CTH)$
CALCULATE SKIN THICKNESS REQUIRED FOR SHEAR STRESS DUE TO AERODYNAMIC LIFT
 $TKSAL = 2.0 * FZ(1) / (PI * DSUBB * FSA)$
CALCULATE SKIN THICKNESS REQUIRED FOR HOOP STRESS. IT IS ASSUMED THE PRESSURE DIFFERENTIAL WILL NOT EXCEED 4 PSI
 $PDIFF = 4.0$
CALCULATE EQUIVALENT PANEL RADIUS OF CURVATURE.
 $ENSUBR = 0.6 * (1.7 - 2.0 * ALB * C7 / DSUBB)$
 $APRC = 0.5 * DSUBB * ENSUBR / CTH$
 $TKHOOP = FS * PDIFF * APRC / FTA$
SELECT LARGEST VALUE OF SKIN THICKNESS. USE HALF THIS VALUE AS FACE THICKNESS.
 $TF = 0.5 * AMAX1(TKMCL, TKCIL, TKSAL, TKHOOP)$
NOW COMPARE THE THEORETICAL FACE THICKNESS WITH THE MINIMUM ALLOWABLE FACE THICKNESS.
THE FACE THICKNESS MUST BE INCREASED TO MINIMUM ALLOWASD0304
FACE THICKNESS.
IF $(TF * LT * TMINC) TF = TMINC$
NEXT, STANDARD GAUGES ARE TO BE USED IF KTFSTD IS 1.
IF $(KTFSTD * NE. 1) GO TO 239$

SWD0270
SWD0271
SWD0272
SWD0273
SWD0274
SWD0275
SWD0276
SWD0277
SWD0278
SWD0279
SWD0280
SWD0281
SWD0282
SWD0283
SWD0284
SWD0285
SWD0286
SWD0287
SWD0288
SWD0289
SWD0290
SWD0291
SWD0292
SWD0293
SWD0294
SWD0295
SWD0296
SWD0297
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SWD0337
SWD0338
SWD0339
SWD0340
SWD0341
SWD0342
SWD0343
SWD0344
SWD0345
SWD0346

JF = 1
237 IF (TSTD(JF) .GT. TF) GO TO 238
IF (JF .GT. 30) GO TO 98
234 JF = JF + 1
GO TO 237
238 TF = TSTD(JF)
239 CONTINUE
FFAX(1) = ANFIMX / (2. * TF)
FFC(1) = (PDESMX * DSUBB) / (4. * TF * CTH)
C
C
CALCULATE CELL SIZE.
CELLWD = SQRT((2. * TF**2*E)/(C8*AMAX1(FFAX(1),FFC(1))))
NCELL = CELLWD * 16.
CELSIZ(1) = FLOAT(NCELL) * .0625
IF (CELSIZ(1) .GT. 1.) CELSIZ(1) = 1.
THE CORE USED IN THIS ANALYSIS IS ASSUMED TO HAVE A HEXAG
ONAL SHAPE. INITIALIZE DIMENSIONAL DATA.
C
C
BETADG = 60.0
BETA = 0.017453 * BETADG
SBETA = SIN(BETA)
W = 0.5 * (A + B)
R = B/A
B = A
A = CELSIZ / (2.0 * SBETA)
CBETA = COS(BETA)
SET FIRST TRIAL RIBBON THICKNESS
TRIBN = AMAX1(TRBMIN:NF) * .0005
IF (KTRSTD .NE. 1) GO TO 224
JG = 1
222 IF (TRBSTD(JG) .GE. TRIBN) GO TO 223
JG = JG + 1
IF (JG .GT. 8) GO TO 98
GO TO 222
223 TRIBN = TRBSTD(JG)
224 CONTINUE
XL=ALB
ASSUME THE RIBBON DIRECTION : PARALLEL TO THE VEHICLE
C

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252 CONTINUE
253 CONTINUE
IF ( H/W .GT. 5.0) GO TO 227
Y = (9.3 / (H/W)) - (2.1 / (H/W)**2)
IF ( H/W .LT. 0.5) Y = 11.0
227 CONTINUE
G1 = GC1 * (1.0 + .01 * Y)
G2 = GC2 * (1.0 + .01 * Y)
P = PDESMN
C THIS BAY HAS PASSED A PANEL STABILITY TEST ON THE
C WINDWARD SIDE. NEXT, CHECK LEEWARD SIDE.
CALL INSTBL
IF (KG .EQ. 1) GO TO 98
IF (CRG .GE. ANFIMX) GO TO 254
IF ( H .GT. 2.0) GO TO 232
H = H + .125
GO TO 253
232 CONTINUE
IF (KTFSTD .EQ. 1) GO TO 234
TF = 1.1 * TF
GO TO 239
254 CONTINUE
C THE DESIGN HAS PASSED THE PANEL STABILITY TEST. THE
C PANEL IS NOW CHECKED FOR WRINKLING, USING THE ANALYSIS BY
C BURNS IN LOCKHEED TECHNICAL REPORT 6-62-64-17, DEC. 1964,
C PAGES 3-3 AND 3-4.
241 AFAC = (2.496 * TF / H ) * ((EFACE * AMINI(A,B)) /
1((GCORE * TRIBN)**0.33333)
IF (AFAC .LT. 1.) GO TO 243
THIN CORE
AFWRIN = 0.93 * EFACE * ((GCORE * TRIBN * TF) /
1(EFACE * AMINI(A,B) * H ))**0.5
GO TO 244
THICK CORE
C 243 AFWRIN = 0.83 * EFACE * ((GCORE * TRIBN)/(EFACE * AMINI(A,B)))
1**0.66667
244 CONTINUE
C COMPARE WRINKLING ALLOWABLE AND APPLIED STRESSES. IF

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SWD0386
SWD0387
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SWD0423


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C      WRINKLING IS PREDICTED, INCREASE RIBBON THICKNESS.
      IF (AFWRIN - AMAX1(FFAX(I),FFC(I)))246, 247, 247
246 CONTINUE
C      DESIGN IS INADEQUATE. INCREASE RIBBON THICKNESS. RETEST.
      IF (KTRSTD *NE. 1) GO TO 271
      IF (JG *GT. 8) GO TO 98
      JG = JG + 1
      TRIBN = TRBSTD(JG)
      GO TO 241
271 TRIBN = TRIBN + .00001
      GO TO 241
247 CONTINUE
C      PANEL DESIGN FOR THIS BAY IS NOW STRUCTURALLY ADEQUATE.
C      COMPUTE BAY DIMENSIONS NEEDED IN WEIGHT CALCULATIONS.
C      DSUBB IS OUTSIDE BASE DIAMETER, D(I) IS THE UPPER OUTSIDE
C      DIAMETER, DBNSD IS INSIDE BASE DIAMETER AND DINSD IS UPPER
C      INSIDE DIAMETER.
      D(I) = DSUBB - (C2)*(ALB)
      DINSD = D(I) - 2. * ((H - TF) / CTH)
      DBNSD = DSUBB - 2. * ((H - TF) / CTH)
C      CALCULATE WEIGHT OF INNER FACE.
      WTINFC = (ALB * C5) * (DBNSD + DINSD) * TF *
1 ( RHO / 2. )
C      CALCULATE WEIGHT OF OUTER FACE
      WTOTFC = (ALB * C5) * (DSUBB + D(I)) * TF *
1 ( RHO / 2. )
      WFACE(I) = WTINFC + WTOTFC
C      CALCULATE CORE WEIGHT
      VCORE = (PI/2. ) * (DBNSD + DINSD) * (ALB / CTH) *
1 (H - TF)
      ROCORE = (TRIBN * RHO * (1. +R)) / (A * SBETA * (R + CBETA))
      WTCORE(I) = VCORE * ROCORE
C      MAKE AN ALLOWANCE FOR THE WEIGHT OF BONDING MATERIAL
C      ASSUME ONE OUNCE PER SQ. FT. PER FACE
      WBOND = (PI/2. ) * (DBNSD + DINSD) * (ALB / CTH)

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 SWD0462

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1 * 0.125 / 144.0
C MAKE AN ALLOWANCE FOR CIRCUMFERENTIAL SPLICE WEIGHT.
WSPLCE(I) = PI * (D(I) + DINS) * (TF * (3.5 + H + TF)) * RHO
C THE REQUIRED MOMENT OF INERTIA OF A STIFFENING RING IS
C CALCULATED.
CALL IREQ
C STIFFENING RING FOR THE BAY IS DESIGNED USING STANDARD
C SKIN GAUGES AND INPUT NORMALIZED RING CROSS-SECTION.
K = 0
256 K = K + 1
IF (K .GT. 29) GO TO 98
CALL RING
C CHECK RING TO SEE IF MOMENT OF INERTIA IS ADEQUATE, AND
C IF OUTSTANDING FLANGE WILL BUCKLE.
CALL RSTRES
IF (FCFB .LT. FRING) GO TO 256
IF (AIRING .GT. AIREQ) GO TO 257
GO TO 256
257 CONTINUE
WRING(I) = AST * (DINS - 2. * DD / CTH) * PI * RHO
WSEG(I) = WFACE(I) + WTCORE(I) + WBOND + WRING(I) + WSPLCE(I)
VSEG(I) = (PI*ALB/12.)*(DSUBB**2 + DSUBB*D(I) + D(I)**2)
WTDEX(I) = WSEG(I) * 1728. / VSEG(I)
ALOPT(I) = ALB
TFACE(I) = TF
TRING(I) = TSTD(K)
TCORE(I) = H
SLOPT = SLOPT + ALB
SUMAL = SUMAL + ALOPT(I)
TRBN(I) = TRIBN
FFS(I) = FZ(I) / (PI * DSUBB * TF)
FFB(I) = FS * PDIFF * APRC / (2. * TF)
FFWR(I) = AFWRIN
FFDMP(I) = (2. * TF**2 * EFACE) / (C8 * CELSIZ(I)**2)
C
C BAY WEIGHT IS ADDED TO PREVIOUS BAY WEIGHTS FOR THIS
C FRUSTUM.
C WCONE = WCONE + WSEG(I)

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SWD0464
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SWD0500

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C      BAY VOLUME WHICH IS USEFUL FOR PAYLOAD IS CALCULATED
C      NEXT. TWO RING HEIGHTS AND TWO PERCENT OF THE BASE DIAMETER
C      ARE SUBTRACTED FROM THE DIAMETERS OF THE CONICAL FRUSTUM IN
C      ORDER TO DETERMINE THIS USEFUL VOLUME.
C      USEFUL BAY VOLUME IS ADDED TO THE SUM OF PREVIOUS BAY
C      VOLUME FOR THIS FRUSTUM.
C      DUSE1 = DSUBB - 2. * AOT * TRING(I) - .02 * DBASE
C      DUSE2 = D(I) - 2. * AOT * TRING(I) - .02 * DBASE
C      VUSE = VUSE + .2617994 * ALOPT(I)*(DUSE1**2 + DUSE1*DUSE2
C      1      DUSE2**2)/1728.
C
C      BASE DIAMETER OF THE NEXT BAY IS SET EQUAL TO THE TOP
C      DIAMETER OF THE CURRENT BAY.
C      DSUBB = D(I)
C
C      THE NUMBER OF BAYS IN THIS FRUSTUM ARE COUNTED.
C      NBAY = NBAY + 1
C      A CHECK IS MADE TO DETERMINE IF THIS IS THE LAST BAY
C      OF THE FRUSTUM.
C      IF( ALB. GE. ALMX2) GO TO 362
C      THE PROGRAM GOES ON TO DESIGN THE NEXT BAY.
C      GO TO 101
C      362 CONTINUE
C
C      THE LAST BAY OF THE FRUSTUM HAS BEEN DESIGNED. WEIGHT
C      OF THE TOP RING OF THE FRUSTUM IS INCREASED TO ALLOW FOR
C      ATTACHMENT PROVISIONS.
C      WRING(I) = 2. * WRING(I)
C      DATA ON THIS FRUSTUM ARE WRITTEN OUT FOR THE SUMMARY
C      REPORT.
C      WRITE (6,313) NF,DMX,DMIN,THTA(NF),
C      1ALCONE,ELMIN(NF),TMNC(NF),NBAY,VUSE,WCONC
C      DMX = DMN(NF)
C      GROSS VOLUME OF THE FRUSTUM IS CALCULATED AND ADDED TO
C      THE GROSS VOLUME OF PREVIOUS FRUSTUMS.
C      VGROSS = 0.2617994*ALCONE*(DBASE**2+DBASE*DMIN+DMIN**2)
C      VGROSS = VGROSS + VGROSS

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C SWD0540
C SWD0541 USEFUL VOLUME OF THIS FRUSTUM IS ADDED TO USEFUL VOLUME
C SWD0542 OF PREVIOUS FRUSTUMS.
C SWD0543 VTOT = VTOT + VUSE
C SWD0544 WEIGHT OF THIS FRUSTUM IS ADDED TO WEIGHT OF PREVIOUS
C SWD0545 FRUSTUMS.
C SWD0546 WTOT = WTOT + WCONE
C SWD0547 BASE DIAMETER OF NEXT FRUSTUM IS SET EQUAL TO TOP
C SWD0548 DIAMETER OF THIS FRUSTUM.
C SWD0549 DBASE = DMIN
C SWD0550 TOTAL NUMBER OF BAYS DESIGNED UP TO THIS POINT IS STORED.
C SWD0551 IMX(NF) = I
C SWD0552 PROGRAM GOES ON TO NEXT FRUSTUM.
C SWD0553
C SWD0554
C SWD0555
C SWD0556
C SWD0557 CONTROL HAS BEEN TRANSFERRED TO THIS POINT BECAUSE THE
C SWD0558 INDEX OF ONE OF THE SUBSCRIPTED VARIABLES HAS EXCEEDED THE
C SWD0559 NUMBER PERMITTED BY THE DIMENSION STATEMENT.
C SWD0560 WRITE (6,120)
C SWD0561 WRITE (6,121)
C SWD0562 WRITE (6,614) I, J, JF, JG, K
C SWD0563 IF (KG .EQ. 1) WRITE(6,201)GBAR
C SWD0564 WRITE (6, 501)
C SWD0565 326 CONTINUE
C SWD0566 IMAX = 1
C SWD0567 WRITE(6,314)
C SWD0568 WRITE (6,315) ALTOT,IMAX,VTOT,WTOT
C SWD0569 WRITE (6,606)
C SWD0570 SCAP SURFACE AREA OF THE NOSE CAP IS COMPUTED
C SWD0571 = 2. * PI * RCAP * ALCAP
C SWD0572 NOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL INTEG-
C SWD0573 RITY IS DETERMINED AND COMPARED WITH THICKNESS PREVIOUSLY
C SWD0574 CALCULATED TO MEET THERMAL REQUIREMENTS. THE LARGER OF THE
C SWD0575 TWO THICKNESSES IS USED TO COMPUTE CAP WEIGHT.
C SWD0576 CALL TNOSSST
C SWD0577 TCAP = AMAX1(TCAPST, TCAPTH)
C SWD0578 NOSE CAP WEIGHT IS COMPUTED.

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C      WCAP      = SCAP* RHO * TCAP
      USEFUL VOLUME OF NOSE CAP IS CALCULATED.
      RUSE      = RCAP - 2. * AOT * TRING(I) - .01 * DBASE
      HUSE      = ALCAP - 2. * AOT * TRING(I) - .01 * DBASE
      VCAP      = 1.0471976*HUSE*HUSE*(3.*RUSE - HUSE)/1728.
      GROSS VOLUME OF CAP IS COMPUTED AND ADDED TO GROSS
      VOLUME OF THE FRUSTUMS.
      VGROSS   = 1.0471976*ALCAP*ALCAP*(3.*RCAP - ALCAP)
      VGGROSS  = (VGGROSS + VGROSS)/ 1728.
      USEFUL VOLUME OF CAP IS ADDED TO USEFUL VOLUME OF CONE
      TO OBTAIN TOTAL USEFUL VOLUME OF FAIRING.
      VTOT     = VTOT + VCAP
      TOTAL LENGTH OF FAIRING IS OBTAINED BY SUMMING CONE AND
      CAP LENGTHS.
      TOTAL FAIRING WEIGHT IS THE SUM OF CAP WEIGHT AND CONE
      WEIGHT.
      WTOT     = WTOT + WCAP
      DATA REPORT IS WRITTEN OUT.
      WRITE (6,82)
      WRITE (6,511) PDSPH
      WRITE (6,67)  TMINN
      WRITE (6,96)  TMPMAX
      WRITE (6,89)  TCAPST
      WRITE (6,90)  TCAPTH
      WRITE (6,91)  TCAP
      WRITE (6,87)  RCAP
      WRITE (6,84)  ALCAP
      WRITE (6,88)  SCAP
      WRITE (6,316) VCAP
      WRITE (6,94)  WCAP
      WRITE (6,92)
      WRITE (6,85)  ALTOT
      WRITE (6,86)  VTOT
      WRITE (6,329) VGGROSS
      WRITE (6,95)  WTOT
      IF AN INTEGER WAS READ IN FOR KEY, DETAILED DESIGN
      INFORMATION IS WRITTEN OUT FOR THE FAIRING.
      IF (KEY .LT. 1) GO TO 10

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 SWD0616

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327 CONTINUE
  WRITE (6, 501)
  WRITE (6,606)
  WRITE (6,51)
  WRITE (6, 14)
  14 FORMAT (59X, 10HRING DATA ///)
  GO TO ( 1615, 1616, 1617 ), IKIND
1615 WRITE (6,2611)
2611 FORMAT( 23X,4HTYPE , 61X, 11HANGLE )
  GO TO 2625
1616 WRITE (6,2612)
2612 FORMAT( 23X, 4HTYPE , 61X, 11HZEE SECTION )
  GO TO 2625
1617 WRITE (6,2613)
2613 FORMAT(23X, 4HTYPE , 61X, 11HAT SECTION )
2625 CONTINUE
  WRITE (6,1620) BOT , AOT, COT, FCFB
1620 FORMAT( 23X, 12HBASE LEG B/T , 53X, 11HBOT
  1 23X, 15HUPRIGHT LEG B/T , 50X, 11HAOT
  2 23X, 19HOUTSTANDING LEG B/T , 46X, 11HCOT
  3 23X, 27HFLANGE BUCKLING STRESS, PSI , 38X, 11HFCFB
  WRITE (6, 22)
  22 FORMAT (55X, 18HSANDWICH FACE DATA //)
  GO TO (671,672,673), MATF
671 WRITE (6,676)
676 FORMAT (23X, 13HFACE MATERIAL , 36X, 24H2024-T4 ALCLAD ALUMINUM )
  GO TO 679
672 WRITE (6,677)
677 FORMAT (23X,13HFACE MATERIAL , 42X, 17H2024-T4 ALUMINUM )
  GO TO 679
673 WRITE (6,678)
678 FORMAT (23X,13HFACE MATERIAL , 42X, 17H7075-T6 ALUMINUM )
679 CONTINUE
  WRITE (6,501)
  WRITE (6,167)
  WRITE (6,168)
  WRITE (6,169)
  WRITE (6,607)

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SWD0650
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SWD0652
SWD0653
SWD0654

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NF          = 1
IW          = 1
I1         = 1
I2         = 46
451 CONTINUE
   IF (I2 .GT. IMAX) I2 = IMAX
   DO 453 I = I1, I2
   ENFIMN(I) = - ENFIMN(I)
   ENFIMX(I) = - ENFIMX(I)
   WRITE (6,58) NF, IW, D(I), ALOPT(I), TFACE(I), TRBN(I), TRING(I), SWD0665
1 WFACE(I), WTCORE(I), WRING(I), WTDEX(I), TCORE(I), WSEG(I),
2CELSIZ(I)
   IW       = IW + 1
   IF (IMX(NF) .GT. I) GO TO 453
   IW       = I
   NF       = NF + 1
453 CONTINUE
   IF (I2 .GE. IMAX) GO TO 457
   I1      = I2 + 1
   I2      = I2 + 54
   WRITE (6, 501)
   WRITE (6,606)
   GO TO 451
457 CONTINUE
   WRITE (6, 501)
   WRITE (6,806)
   WRITE (6,807)
   WRITE (6,808)
   WRITE (6,809)
   NF      = I
   IW      = I
   I1      = I
   I2      = 46
461 CONTINUE
   IF (I2 .GT. IMAX) I2 = IMAX
   DO 463 I = I1, I2
   WRITE (6,508) NF, IW, ENFIMN(I), ENFIMX(I), FFA(X(I), FFC(I),
1 FFS(I), FFB(I), FFWR(I), FCA, FFDMP(I)
   IW     = IW + 1

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SWD0655
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 SWD0731

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IF (IMX(NF) .GT. I) GO TO 463
IW = 1
NF = NF + 1
463 CONTINUE
IF (I2 .GE. IMAX) GO TO 467
I1 = I2 + 1
I2 = I2 + 54
WRITE (6, 501)
WRITE (6, 606)
GO TO 461
467 CONTINUE
C
C IF 2 WAS READ IN FOR KEY, DETAILED LOADS INFORMATION
IS WRITTEN OUT.
IF (KEY .NE. 2) GO TO 10
WRITE (6, 501)
WRITE (6, 606)
WRITE (6, 612)
WRITE (6, 608)
WRITE (6, 609)
WRITE (6, 610)
SUMAL = 0.
IW = 1
NF = 1
I1 = 1
I2 = 46
THETA = 0.0174532925*THTA(NF)
CTH = COS(THETA)
STH = SIN(THETA)
C2 = 2.*STH/CTH
C3 = 0.3/CTH
DSUBB = DBAS
ALB = ALOPT(1)
LPFL = 1
471 IF (IMAX .LT. I2) I2 = IMAX
DO 473 I = I1, I2
CALL PRESUR
LPFL = 2
ANFIAX = 0.5*(ENFIMN(I) + ENFIMX(I))/FS
    
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ANFIB = ANFIAX - ENFIMN(I)/FS
AXLOAD =--ANFIAX*DSUBB*.94248/C3
BEND =--0.23562*ANFIB*DSUBB**2/C3
WRITE (6,611) NF, IW, DSUBB, D(I), ALOPT(I), SUMAL, PDES MX, PDESMN,
1 AXLOAD, FZ(I), BEND
IW = IW + 1
SUMAL = SUMAL + ALOPT(I)
DSUBB = D(I)
ALB = ALOPT(I+1)
IF (IMX(NF) .GT. I) GO TO 473
IW = 1
NF = NF + 1
THETA = 0.0174532925*THTA(NF)
CTH = COS(THETA)
STH = SIN(THETA)
C2 = 2.*STH/CTH
C3 = 0.3/CTH
473 CONTINUE
IF (IMAX .LE. I2) GO TO 477
I1 = I2 + 1
I2 = I2 + 54
WRITE (6, 501)
WRITE (6,606)
GO TO 471
477 CONTINUE
WRITE (6,613) DSUBB, SUMAL, AXLD CP, FSBZCP, BND CAP
GO TO 10
51 FORMAT (48X,36H DESIGN DETAILS OF CONICAL FRUSTUMS ///)
54 FORMAT (54X,24H FAIRING GEOMETRY //)
58 FORMAT(1X,I2,I1H-,I2,4X,F6.1,F8.1,F10.4,F9.5,F9.4,F11.2,
1 F11.2, F9.2, F11.4, F11.4, F11.1, F10.4)
63 FORMAT (48X,33H CONSTRAINTS ON FAIRING DESIGN //)
64 FORMAT (23X,77H MINIMUM BAY LENGTH CONSIDERED, IN,
1 ALMIN = ,F8.3)
66 FORMAT (23X,77H MINIMUM ALLOWABLE THICKNESS OF CONE SKIN, IN,
1 TMINC = ,F8.4)
67 FORMAT (23X,77H MINIMUM ALLOWABLE THICKNESS OF NOSE CAP SKIN, IN,
1 TMINN = ,F8.4 )
68 FORMAT (48X,36H SPECIFIED DESIGN PRESSURES //)

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SWD0732
SWD0733
SWD0734
SWD0735
SWD0736
SWD0737
SWD0738
SWD0739
SWD0740
SWD0741
SWD0742
SWD0743
SWD0744
SWD0745
SWD0746
SWD0747
SWD0748
SWD0749
SWD0750
SWD0751
SWD0752
SWD0753
SWD0754
SWD0755
SWD0756
SWD0757
SWD0758
SWD0759
SWD0760
SWD0761
SWD0762
SWD0763
SWD0764
SWD0765
SWD0766
SWD0767
SWD0768
SWD0769
SWD0770

69 FORMAT (23X,77H DESIGN PRESSURE ON WNDWRD SIDE OF CONE, (SAFETY FASWD0771
 1CTOR=1.4), PSI PDESMX = ,F8.3) SWD0772
 71 FORMAT (45X,42H AERODYNAMIC DATA USED IN COMPUTING LOADS //) SWD0773
 72 FORMAT (23X,77H DYNAMIC PRESSURE, LBS./SQ. FT.
 1 QBAR = ,F8.2) SWD0774
 73 FORMAT (23X,77H MACH NUMBER AT DESIGN DYNAMIC PRESSURE
 1 AMACH = ,F8.3) SWD0775
 74 FORMAT (23X,77H ANGLE OF ATTACK AT DESIGN DYNAMIC PRESSURE, DEGREES
 1S ALPHA = ,F8.2) SWD0776
 75 FORMAT (48X, 36H COMPUTED AERODYNAMIC LOADS DATA //) SWD0777
 76 FORMAT (23X,77H PRESSURE COEFFICIENT ON CONE AT ZERO ANGLE OF ATTASWD0781
 1CK CPO = ,F8.4) SWD0782
 82 FORMAT (57X,16H NOSE CAP DESIGN //) SWD0783
 83 FORMAT (23X,77H LENGTH OF CONICAL SECTION, IN.
 1 ALCONE = ,F8.2) SWD0784
 84 FORMAT (23X,77H LENGTH OF NOSE CAP, IN.
 1 ALCAP = ,F8.2) SWD0785
 85 FORMAT (23X,77H TOTAL LENGTH OF FAIRING, IN.
 1 ALTOT = ,F8.2) SWD0786
 86 FORMAT (23X,77H USEFUL VOLUME OF FAIRING, CU. FEET
 1 VTOT = ,F8.2) SWD0787
 87 FORMAT (23X,77H NOSE CAP RADIUS, IN.
 1 RCAP = ,F8.3) SWD0788
 88 FORMAT (23X,77H NOSE CAP SURFACE AREA, SQ. IN.
 1 SCAP = ,F8.2) SWD0789
 89 FORMAT (23X,77H NOSE CAP SKIN THICKNESS REQUIRED FOR STRUCTURAL IN
 1TEGRITY, IN. TCAPST = ,F8.3) SWD0790
 90 FORMAT (23X,77H NOSE CAP SKIN THICKNESS REQUIRED FOR THERMAL REASOS
 1NS, IN. TCAPTH = ,F8.3) SWD0791
 91 FORMAT (23X,77H NOSE CAP SKIN THICKNESS USED TO CALCULATE WEIGHT,
 1IN. TCAP = ,F8.3) SWD0792
 92 FORMAT (44X,43H TOTAL LENGTH, VOLUME AND WEIGHT OF FAIRING //) SWD0793
 93 FORMAT (23X,77H WEIGHT OF CONICAL SECTION, LBS.
 1 WCONE = ,F8.2) SWD0794
 94 FORMAT (23X,77H WEIGHT OF NOSE CAP, LBS.
 1 WCAP = ,F8.2 ///) SWD0795
 95 FORMAT (23X,77H TOTAL WEIGHT OF FAIRING, LBS.
 1 WTOT = ,F8.2) SWD0800
 SWD0801
 SWD0802
 SWD0803
 SWD0804
 SWD0805
 SWD0806
 SWD0807
 SWD0808

96 FORMAT (23X,77H MAXIMUM ALLOWABLE TEMPERATURE OF SKIN, DEG. F. SWD0809
 1 TMPMAX = ,F8.1) SWD0810
 120 FORMAT (23X,80H DESIGN OF THE CONICAL SECTION OF THE FAIRING HAS NSWD0811
 10T BEEN COMPLETED. THE I, J,) SWD0812
 121 FORMAT (23X,80H OR K INDEX HAS EXCEEDED THE MAXIMUM PERMITTED BY TSWD0813
 1HE DIMENSION STATEMENT.) SWD0814
 131 FORMAT (5F12.8, 2I6) SWD0815
 132 FORMAT (2I6,4F12.8,I12) SWD0816
 162 FORMAT (23X,77H AMBIENT PRESSURE AT DESIGN CONDITIONS, PSI SWD0817
 1 PSTAT = ,F8.3 ///) SWD0818
 163 FORMAT (23X,77H INTERNAL-TO-AMBIENT PRESSURE DIFFERENCE AT DESIGN SWD0819
 1CONDITIONS,PSI DELTAP = ,F8.3) SWD0820
 164 FORMAT (23X,77H CONICAL SECTION SKIN THICKNESS REQUIRED FOR THERMASWD0821
 1L REASONS, IN. TCONTH = ,F8.4 ///) SWD0822
 165 FORMAT (77X, 9H WINDWARD,27X,8H LEEWARD) SWD0823
 167 FORMAT (128H FRUSTUM SHELL BAY FACE RIBBON RING SWD0824
 1 TOTAL RING WEIGHT SHELL TOTAL SWD0825
 2 CELL) SWD0826
 168 FORMAT (128H -BAY O.D. LENGTH GAUGE GAUGE SWD0827
 1 FACE WT. CORE WT. WT. INDEX THICKNESS BAY WT. SWD0828
 2 WIDTH) SWD0829
 169 FORMAT (128H NO. (IN) (IN) (IN) (IN) SWD0830
 1 (LB) (LB) (LB/CU FT) (IN) (LB) SWD0831
 2 (IN)) SWD0832
 170 FORMAT (23X,77H DESIGN PRESSURE ON LEEWRD SIDE OF CONE (SAFETY FASWD0833
 1CTOR=1.4), PSI PDESMN = ,F8.3) SWD0834
 178 FORMAT (14X,103H RADIUS OF THE NOSE CAP IS ZERO. FOR THIS CASE THESWD0835
 1 HEAT TRANSFER EQUATIONS ARE NOT VALID. THEREFORE, NO/ 9X,70H THERSWD0836
 2MAL CONSTRAINTS HAVE BEEN IMPOSED ON THE DESIGN OF THIS FAIRING. SWD0837
 3 ///) SWD0838
 201 FORMAT (20X,52HCASE TERMINATED SINCE GBAR IS OUTSIDE RANGE. GBAR =SWD0839
 1 E11.4) SWD0840
 206 FORMAT (23X,77H CHANGE IN PRESSURE COEFFICIENT DUE TO ANGLE OF ATTSWD0841
 1ACK CPA = ,F8.4) SWD0842
 307 FORMAT (56X,18H AERODYNAMIC LOADS //) SWD0843
 308 FORMAT (48X,34H CONSTRAINTS ON DESIGN OF FRUSTUMS //) SWD0844
 309 FORMAT (48X,36H DESIGN SUMMARY FOR FRUSTUM SECTION //) SWD0845
 310 FORMAT (20X, 92H FRUS-- LARGESWD0846
 1E SMALL HALF LENGTH MIN. BAY MIN. NO. USEFUSWD0847

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2L WEIGHT )
311 FORMAT (20X, DIASWD0848
1. DIA. ANGLE 92H TUM DIASWD0849
2E ) VOLUMSWD0850
312 FORMAT (20X, SWD0851
1) (IN) (DEG) (IN) 92H NO. (INSWD0852
2) (LB) ) (IN) BAYS (CU FTSWD0853
313 FORMAT(21X,I4, F10.2,F10.2,F9.2,F9.1,F10.1, SWD0854
1 F10.4,I7,F11.2,F11.2) SWD0855
314 FORMAT (57X, 6H ----,24X,25H SWD0856
315 FORMAT (46X,6HTOTALS,F11.1,23X,I4,2F11.2 ) SWD0857
316 FORMAT (23X,77H USEFUL VOLUME OF NOSE CAP, CU. FT. SWD0858
1 VCAP = ,F8.2 ) SWD0859
329 FORMAT (23X,77H TOTAL VOLUME OF FAIRING, CU. FEET SWD0860
1 VGROSS = ,F8.2 ) SWD0861
365 FORMAT (3F12.8,35X,I1) SWD0862
501 FORMAT (1H1) SWD0863
508 FORMAT (1X,I2,I1H-,I2,4X,F8.2,4X,F8.2,5X,F7.0,F9.0,2F10.0,5X,F9.0, SWD0864
1 4X,F7.0,5X,F7.0) SWD0865
511 FORMAT (23X,77H DESIGN PRESSURE ON NOSE CAP, PSI SWD0866
1 PDSPH = ,F8.3 ) SWD0867
550 FORMAT (42X,48HDATA REPORT FOR OPTIMIZED NOSE FAIRING STRUCTURE /)SWD0868
555 FORMAT (59X,I2HCASE NUMBER ,I3 //) SWD0869
606 FORMAT (1X, //) SWD0870
607 FORMAT (1X, //) SWD0871
608 FORMAT (10X,I12H FRUSTRUM BAY BASE BAY TOP BAY DISTANCE SWD0872
1 DES. PRES. DES. PRES. AXIAL SHEAR BENDING )SWD0874
609 FORMAT (10X,I12H -BAY DIA. DIA. LENGTH FROM BASESWD0875
1 WNDWRD LEWRD LOAD LOAD MOMENT ;SWD0876
610 FORMAT (10X,I12H NO. (IN) (IN) (IN) SWD0877
1 (PSI) (PSI) (LBS) (IN-LBS) //)SWD0878
611 FORMAT (10X,I2,I1H-,I2,F12.1,F10.1,F9.1,F11.1,F12.2,F13.1, SWD0879
1 F12.1,F14.1)
612 FORMAT (54X,24H DETAILED LOADS DATA ///) SWD0880
613 FORMAT (17H TANG. PT.,F10.1,19X,F11.1,25X,F13.1,F12.1, SWD0881
1 F14.1) SWD0882
614 FORMAT (22X,93H DESIGN OF THE FAIRING HAS NOT BEEN COMPLETED. ONE SWD0883
1 OF THE SUBSCRIPTED VARIABLES HAS EXCEEDED /35X,20H THE MAXIMUM VALSWD0885

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2UE. /11X,13H I(MAX) = 400 ,6X,12H J(MAX) = 29 ,7X,13H JF(MAX) = 29SWD0886
3 ,6X,13H JG(MAX) = 8 ,7X,13H K(MAX) = 29 ,7X,12H M(MAX) = 7 / SWD0887
4 16X,3HI = ,14, 11X,3HJ = ,13,12X,4HJF = ,13,11X,4HJG = ,12, 12X, SWD0888
5 3HK = ,13,12X,3HM = ,12 / ) SWD0889
650 FORMAT (1H1) SWD0890
806 FORMAT( 39X, 28HMAX APPLIED FACE STRESS, PSI , 15X, SWD0891
1 21HALLOWABLE STRESS, PSI // ) SWD0892
807 FORMAT (1X,7HFRUSTUM, 3X, 8HWINDWARD, 4X, 7HLEEWARD, 49X, 4HFACE, SWD0893
1 8X, 4HFACE , 7X, 4HFACE ) SWD0894
808 FORMAT ( 5H -BAY , 5X, 99HLINE LOAD LINE LOAD AXIAL CIRC SWD0895
1 SHEAR BURST WRINKLING YIELD DIMPLING ) SWD0896
809 FORMAT (4H NO. , 7X, 20H(LB/IN) (LB/IN) // ) SWD0897
989 FORMAT(15,6E12,8) SWD0898
990 FORMAT (16, 5F12,4, 16) SWD0899
END SWD0900
$IBFTC C53S2 DECK PTY 001
SUBROUTINE PROPT (MAT, E, AMU, RHO, TMPMAX) PTY 002
GO TO (1,2,3,4,5 ) , MAT PTY 003
1 CONTINUE PTY 004
2 CONTINUE PTY 005
3 CONTINUE PTY 006
4 CONTINUE PTY 007
5 CONTINUE PTY 008
6 CONTINUE PTY 009
7 CONTINUE PTY 010
8 CONTINUE PTY 011
9 CONTINUE PTY 012
10 CONTINUE PTY 013
11 CONTINUE PTY 014
12 CONTINUE PTY 015
13 CONTINUE PTY 016
14 CONTINUE PTY 017
15 CONTINUE PTY 018
16 CONTINUE PTY 019
17 CONTINUE PTY 020
18 CONTINUE PTY 021
19 CONTINUE PTY 022
20 CONTINUE PTY 023

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WRITE (6,143)
143 FORMAT (54X,24H MATERIAL = TITANIUM ///)
E = 16000000.0
AMU = 0.3
RHO = 0.16
TMPMAX = 900.0
RETURN
4 CONTINUE
C STAINLESS STEEL PROPERTIES (MAT = 4)
WRITE (6,144)
144 FORMAT (54X,24H MATERIAL = STEEL ///)
E = 30000000.0
AMU = 0.3
RHO = 0.283
TMPMAX = 1100.0
RETURN
5 CONTINUE
C LOCKALLOY PROPERTIES (MAT = 5)
WRITE (6,145)
145 FORMAT (54X,24H MATERIAL = LOCKALLOY ///)
E = 27000000.0
AMU = 0.3
RHO = 0.076
TMPMAX = 700.0
RETURN
END
$IBFTC C53S3 DECK
SUBROUTINE THERML
C THIS SUBROUTINE COMPUTES SKIN THICKNESSES REQUIRED TO
C KEEP NOSE CAP AND TOP FRUSTUM SKIN TEMPERATURES UNDER TMPMAX.
DIMENSION AT(50),AS(20)
COMMON /TNOS/TMINN,PDSPH,TCAPST,RCAP
COMMON /THRML/ TMPMAX,MAT,TCONTH,TCAPTH,THETA
C THE EQUATIONS AND STORED COEFFICIENTS IN THIS SUBROUTINE
C WERE OBTAINED BY MEANS OF A MULTIPLE REGRESSION ANALYSIS OF
C DATA RESULTING FROM A THERMAL ANALYSIS OF SPHERES FLYING A
C NOMINAL LLSV TRAJECTORY. THE HOTTEST POINT ON THE NOSE CAP
C IS AT THE STAGNATION POINT OF THE SPHERE, AND THE HOTTEST
C POINT ON THE TOP FRUSTUM IS AT THE TANGENCY POINT OF THE

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PTY 024
PTY 025
PTY 026
PTY 027
PTY 028
PTY 029
PTY 030
PTY 031
PTY 032
PTY 033
PTY 034
PTY 035
PTY 036
PTY 037
PTY 038
PTY 039
PTY 040
PTY 041
PTY 042
PTY 043
PTY 044
PTY 045
PTY 046
PTY 047
PTY 048
PTY 049
TML 001
TML 002
TML 003
TML 004
TML 005
TML 006
TML 007
TML 008
TML 009
TML 010
TML 011
TML 012
TML 013

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C SPHERICAL NOSE CAP AND TOP FRUSTUM. THE METHOD USED TO OBTAIN TML 014
 C THE ORIGINAL THERMAL DATA IS DESCRIBED IN LMSC DOCUMENT NO. TML 015
 C TM 54-20-7. TML 016
 C TANGENCY POINT COEFFICIENTS FOR ALUMINUM. TML 017
 DATA AT/ .027903, -.010162, .691498, -.013716, 2.108521, TML 018
 1 .25293, .52256, -.560318, .896866, -.512848, TML 019
 C TANGENCY POINT COEFFICIENTS FOR MAGNESIUM. TML 020
 2 -.176254, -1.175537, 1.972534, -.051082, 2.988892, TML 021
 C TANGENCY POINT COEFFICIENTS FOR TITANIUM. TML 022
 3 -.392456, 1.529938, -1.154861, .88208, -.055298, TML 023
 4 .056461, .062469, .579819, -.018108, 2.341248, TML 024
 5 .206238, .399368, -.336502, .835144, -.355804, TML 025
 C TANGENCY POINT COEFFICIENTS FOR STAINLESS STEEL. TML 026
 6 .194916, 1.457764, -1.19165, -.064727, 1.619629, TML 027
 C TANGENCY POINT COEFFICIENTS FOR LOCKALLOY. TML 028
 7 -.032959, -.851807, .946594, 1.026367, -.943359, TML 029
 8 .352531, 1.183907, .312973, .001026, 1.551514, TML 030
 9 .084351, -.735126, -.335369, .614578, -.366287/ TML 031
 C STAGNATION POINT COEFFICIENTS FOR ALUMINUM. TML 032
 DATA AS/- .014267, 5.291153, -1.559019, -.010812, TML 033
 C STAGNATION POINT COEFFICIENTS FOR MAGNESIUM. TML 034
 1 -.029125, 6.776562, -1.966431, -.033269, TML 035
 C STAGNATION POINT COEFFICIENTS FOR TITANIUM. TML 036
 2 -.001164, 5.252344, -0.926601, -.012582, TML 037
 C STAGNATION POINT COEFFICIENTS FOR STAINLESS STEEL. TML 038
 3 0.009530, 3.505341, -1.058136, -.017291, TML 039
 C STAGNATION POINT COEFFICIENTS FOR LOCKALLOY. TML 040
 4 -.008722, 3.676389, -1.086252, -.000541/ TML 041
 = (SIN(THETA))**2 TML 042
 X2 = 1.0/SQRT(RCAP) TML 043
 X3 = 0.001*TMPMAX*X2 TML 044
 X4 = (0.001 * TMPMAX+ 0.46) ** 4 TML 045
 X5 = X2*S2 TML 046
 X6 = X5*(0.001*TMPMAX) TML 047
 X7 = 1.0/(RCAP**0.2) TML 048
 X8 = .001*TMPMAX*X7 TML 049
 X9 = X7*S2 TML 050
 X10 = 0.001*TMPMAX*X9 TML 051

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L      = 1 + 4*(MAT - 1)
M      = 1 + 10*(MAT - 1)
Y TAN  = AT(M)+X2*AT(M+1)+X3*AT(M+2)+X4*AT(M+3)+X5*AT(M+4)
1      +X6*AT(M+5)+X7*AT(M+6)+X8*AT(M+7)+X9*AT(M+8)+X10*AT(M+9)
Y STG  = AS(L)+X2*AS(L+1)+X3*AS(L+2)+X4*AS(L+3)
TCAPTH = 100.*YSTG/(TMPMAX-70.)
TCONTH = 100.*YTAN/(TMPMAX-70.)
RETURN
END
$IBFTC C53S4 DECK
SUBROUTINE AERO
C      THIS SUBROUTINE COMPUTES PRESSURE COEFFICIENTS AT ZERO
C      ANGLE-OF-ATTACK AND THE CHANGE IN PRESSURE COEFFICIENT DUE
C      TO ANGLE-OF-ATTACK. FOR THE PURPOSE OF COMPUTING THESE
C      COEFFICIENTS, EACH FRUSTUM IS TREATED AS A COMPLETE CONE.
C      FIRST, THE PRESSURE COEFFICIENT AT ZERO ANGLE OF ATTACK
C      IS CALCULATED BY THE METHOD OF SIMON AND WALTER, AIAA JOURNAL,
C      JULY 1963, PP 1696-97. THE METHOD APPROXIMATES EXACT
C      SOLUTIONS WITH A MODIFIED QUADRATIC IN SINE SQUARE OF DELTA
C      WITH COEFFICIENTS BEING FUNCTIONS OF GAMMA AND MACH NUMBER.
COMMON/AERPRS/CPAA,CPOO,LPFL
COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
COMMON/AERLOD/ALPHA,AMACH,NF
IF (THTA(NF)) 30,30,31
30 CPOO = 0.
CPAA   = 0.
GO TO 300
31 CONTINUE
DELTA  = 0.0174532925*THTA(NF)
G=GAMMA=1.4, GAM1=(G+7)/(G+1), GAM2=(G+7)/4, GAM3=((G-1)/4)**2
GAM4=G/2, GAM5=GAM4*GAM1, GAM6=GAM1/2
GAM1=3.5
GAM2=2.1
GAM3=.01
GAM4=.7
GAM5=2.45
GAM6=1.75
EM     = AMACH
IF(EM-1.05)100,1,1

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TML 052
TML 053
TML 054
TML 055
TML 056
TML 057
TML 058
TML 059
TML 060
AER 001
AER 002
AER 003
AER 004
AER 005
AER 006
AER 007
AER 008
AER 009
AER 010
AER 011
AER 012
AER 013
AER 014
AER 015
AER 016
AER 017
AER 018
AER 019
AER 020
AER 021
AER 022
AER 023
AER 024
AER 025
AER 026
AER 027
AER 028
AER 029
AER 030


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1 FINK=DELTA*57.2958
  IF(FINK-50.)3,3,4
4 WRITE (6,5) FINK
5 FORMAT(84HOSUBR RAP-AT ANGLES BEYOND 50 DEG THE CP OBTAINED IS LIKAER
  IELY TO BE ERRONEOUS. DELTA=F6.2,6H, EM0=F6.2,7H, X=F8.2)
  EM=0.
  RETURN
3 SDELT=SIN (DELTA)
  SSDEL=SDELT*SDELT
  GSSDE=1.4*SSDEL
  EMSTR=SQRT ((1.+GSSDE)/(1.-GSSDE))
22 IF(EM-EMSTR)110,2,2
  2 EMS=EM*EM
    EM4=EMS*EMS
    EM6=EM4*EMS
    F1=((EMS-1.)/(EM4*SDELT))+(6./EM6)+GAM2-GAM3
    Q1=1.+(1./EM6)
    Q1=1./EMS
    F1SSD=F1*SDELT
    F2=GAM6*(1.-Q1)*Q
    F3=GAM5*(1.+Q1)*Q
    SQUID=(F2-F1SSD)**2-((F3-F1)*SSDEL)**2
    CPOO = 0.5*((F2+F1SSD)-SQRT(SQUID))

    NEXT, THE INCREASE IN PRESSURE COEFFICIENT (ON THE HIGH
    PRESSURE SIDE OF THE CONE) DUE TO ANGLE OF ATTACK IS
    COMPUTED. THIS CALCULATION IS BASED ON DATA FROM CHART 8
    OF NACA REPORT 1135 AT MACH 1.5, AND A CIRCUMFERENTIAL
    PRESSURE DISTRIBUTION WHICH IS SINUSOIDAL. THE EQUATION
    FOR CPAA IS VALID FOR MACH NUMBERS BETWEEN 1.4 AND 1.6.
    CHECK IF MACH NUMBER LIES BETWEEN 1.4 AND 1.6.

    IF (AMACH - 1.4) 201,202,203
    203 IF (AMACH - 1.6) 202,202,201
    201 WRITE (6,205)
      WRITE (6,204)
      WRITE (6,205)
    205 FORMAT (1H1)

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AER 031
AER 032
AER 033
AER 034
AER 035
AER 036
AER 037
AER 038
AER 039
AER 040
AER 041
AER 042
AER 043
AER 044
AER 045
AER 046
AER 047
AER 048
AER 049
AER 050
AER 051
AER 052
AER 053
AER 054
AER 055
AER 056
AER 057
AER 058
AER 059
AER 060
AER 061
AER 062
AER 063
AER 064
AER 065
AER 066
AER 067
AER 068

C
C
C
C
C
C
C
C

```

204 FORMAT (16X,100MACH NUMBER LIES OUTSIDE THE INTERVAL FROM 1.4 TO AER 069
11.6, VALUES CALCULATED FOR CPA MAY BE INACCURATE. ) AER 070
C AER 071
C AER 072
C AER 073
202 CPAA = (2.03 - 1.20*DELTA)*ALPHA*2.*SIN(DELTA)/COS(DELTA) AER 074
300 CONTINUE AER 075
RETURN AER 076
100 WRITE (6,101) EM AER 077
101 FORMAT(32H0 SUBROUTINE RAP--MACH NO.(F6.2,26H) IS LESS THAN 1 AER 078
1.05 EM0=F6.2,7H, EM1=F6.2,5H, X=F8.2) AER 079
EM#0. AER 080
RETURN AER 081
110 WRITE (6,111) FINK, EM AER 082
111 FORMAT(27H0 SUBROUTINE RAP--DELTA(F6.2,42H) EXCEEDS MAX ANGLE FAER 083
1OR THE MACH NO. USED(F6.2,8H). EM0=F6.2,7H, EM1=F6.2,4H, X=F8.2) AER 084
EM#0. AER 085
RETURN AER 086
END AER 087
$IBFTC C53S5 DECK DIA 001
SUBROUTINE DIAM (XN,DLOC) DIA 002
C INPUT PARAMETERS--XN,DMN(NF) FOR NF = 1 TO NFMAX,THTA(NF) DIA 003
C FOR NF = 1 TO NFMAX,NFMAX,ALTOT DIA 004
C OUTPUT VARIABLES--DLOC DIA 005
COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG DIA 006
COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP, FS,LTMAX,SUMAL, DIA 007
1 QBAR,ALTOT DIA 008
XTOT = ALTOT DIA 009
DO 10 N1 = 1,NFMAX DIA 010
XTOT = XTOT - ALF(N1) DIA 011
DELTA = XN- XTOT DIA 012
IF (DELTA) 10,20,20 DIA 013
10 CONTINUE DIA 014
20 CONTINUE DIA 015
ANGLE = 0.0174532925*THTA(N1) DIA 016
DLOC = DMN(N1) + 2.*DELTA*SIN(ANGLE)/COS(ANGLE) DIA 017
RETURN DIA 018
END DIA 019
$IBFTC C53S6 DECK LDS 001

```

```

SUBROUTINE LOAD
THIS SUBROUTINE COMPUTES AXIAL LOADS, SHEAR LOADS,
BENDING MOMENTS AND THE RESULTING LINE LOADS USING EITHER
THE INPUT PRESSURE DATA OR THAT COMPUTED IN SUBROUTINE AERO.

COMMON /DLOAD/D1,D2,XOD1,XOD2,CP01,CP02,CPA1,CPA2,A3,A4,QB,DP,
1 FSBZ,BND,AXLOD
COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
COMMON/LOADS/CDCAP,CNCAP,XBCAP,FSUBZ,AXLDCP,FSBZCP,BNDCAP,LTRIG,
1 ALCAP
COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP, FS,LTMAX,SUMAL,
1 QBAR,ALTOT
COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
COMMON/AERLOD/ALPHA,AMACH,NF
COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1 CHKWND,CHKLEE,C2
COMMON/PRSLOD/ LTJNCT(10)

IF (LTRIG) 399,399,393
399 CONTINUE
NX = NFMAX
LTJNCT(NX)= 1
XOD(1) = ALCAP/DBAS
XODSTP = (ALCAP + ALF(NFMAX))/DBAS
DO 458 LTX= 2,LTMAX
IF ((XOD(LTX) - XOD(LTX-1)).GT..000001) GO TO 458
NX = NX - 1
LTJNCT(NX)= LTX
XOD(LTX) = XODSTP
XOD(LTX-1)= XODSTP
XODSTP = XODSTP + ALF(NX)/DBAS
458 CONTINUE
XOD(LTMAX)= ALTOT/DBAS
QB = QBAR/144.
A3 = 1.5707963*QB*DBAS
A4 = A3*DBAS

```

LDS 002
LDS 003
LDS 004
LDS 005
LDS 006
LDS 007
LDS 008
LDS 009
LDS 010
LDS 011
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 LDS 073
 LDS 074
 LDS 075
 LDS 076
 LDS 077
 LDS 078

DP = DELTAP
 SINCE THIS IS THE FIRST BAY, THE BENDING MOMENT, AXIAL
 LOAD AND SHEAR LOAD AT THE BASE OF THE FAIRING ARE COMPUTED
 USING THE PRESSURE DATA FOR THE ENTIRE FAIRING.
 THE LOADS CONTRIBUTED BY THE CONICAL FRUSTUMS ARE
 COMPUTED FIRST.

```

LTRIG = 1
X = 0.
AXLOAD = 0.
BEND = 0.
FSUBZ = 0.
EL = 0.
N1 = 1
LTMX1 = LTMX + 1
LTR1 = 1
DO 463 N1 = 1, NFMX
LTR2 = LTMX - LTJNCT(N1)
THETA = 0.0174532925*THTA(N1)
C2 = 2.*SIN(THETA)/COS(THETA)
DO 371 LTR= LTR1, LTR2
LT = LTMX1 - LTR
XN = XOD(LT)*DBAS
CALL DIAM (XN,D1)
XN = XOD(LT-1)*DBAS
CALL DIAM (XN,D2)
XOD1 = XOD(LT)
XOD2 = XOD(LT-1)
CPO1 = CPO(LT)
CPO2 = CPO(LT-1)
CPA1 = CPA(LT)
CPA2 = CPA(LT-1)
CALL DL0D
EL = EL+X
X = (XOD1-XOD2)*DBAS
AXLOAD = AXLOAD + AXLOAD
FSUBZ = FSUBZ + FSBZ
BEND = BEND + EL*FSBZ + BND
    
```

C
 C
 C
 C
 C
 C
 C

```

37) CONTINUE          = LTR2 + 1
LTR1
463 CONTINUE
C
C   THE LOADS CONTRIBUTED BY THE SPHERICAL NOSE CAP ARE
C   ADDED TO THE FRUSTUM LOADS.
C
AC      = .7853982*DMN(NFMAX)**2
EL      = EL + X
IF (CDCAP) 435.435.436
435 CONTINUE
X      = DMN(NFMAX)/C2
XBCAP  = X/3.
CZALFA = 2.03 - 1.2*THETA
CNCAP  = CZALFA
CPSTG  = (.166.92158*AMACH**7./((7.*AMACH**2-1.)*2.5-1.)) /
        (0.7*AMACH**2)
1
STHTA  = SIN(THETA)
CDCAP  = CPSTG*(1.+STHTA**2)/2.
436 CONTINUE
XBAR   = XBCAP
AXLOD  = AC*(CDCAP*QB - DELTAP)
FSBZ   = AC*CNCAP*ALPHA*QB
AXLDCP = AXLOD
FSBZCP = FSBZ
BNDCAP = FSBZ*XBAR
AXLOAD = AXLOD + AXLOAD
BEND   = BEND + (EL + XBAR)*FSBZ
FSUBZ  = FSBZ + FSBZ
C
C   PARAMETERS ARE INITIALIZED FOR USE IN COMPUTING UPPER
C   BAY LOADS.
C
LFLG   = 1
THETA  = 0.0174532925*THTA(NF)
C3     = 0.3 / COS(THETA)
C2     = 2.*SIN(THETA)/COS(THETA)
C21    = C2

```

```

LDS 079
LDS 080
LDS 081
LDS 082
LDS 083
LDS 084
LDS 085
LDS 086
LDS 087
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LDS 108
LDS 109
LDS 110
LDS 111
LDS 112
LDS 113
LDS 114
LDS 115
LDS 116

```

```

CPO2 = CPO(LTMAX)
CPA2 = CPA(LTMAX)
XOD2 = XOD(LTMAX)
D2 = DBAS
LT = LTMAX - 1
GO TO 386
393 CONTINUE
XODB = (ALTOT - SUMAL) / DBAS
GO TO (445, 446, 447), LFLG
446 CONTINUE
LFLG = 1
BNDLB = BEND
XODLB = XOD2
CPOLB = CPO2
CPALB = CPA2
D2LB = D2
LTBL = LT
FZLB = FSUBZ
AXLLB = AXLOAD
GO TO 445
447 CONTINUE
BEND = BNDLB
FSUBZ = FZLB
AXLOAD = AXLLB
LFLG = 1
XOD2 = XODLB
CPO2 = CPOLB
CPA2 = CPALB
D2 = D2LB
LT = LTBL
448 CONTINUE

```

SINCE THIS IS AN UPPER BAY, THE LOADS AT THE BASE OF THIS BAY ARE DETERMINED BY FIRST COMPUTING THE LOADS PRODUCED BY THE PRESSURE PROFILE BETWEEN THIS BAY AND THE PREVIOUS BAY, AND THEN SUBTRACTING THESE COMPUTED LOADS FROM THE LOADS OF THE PREVIOUS BAY.

XOD1 = XOD2

LDS 117
LDS 118
LDS 119
LDS 120
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LDS 188
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LDS 190
LDS 191
LDS 192
LDS 193

```

CP01 = CP02
CPA1 = CPA2
D1 = D2
IF (XOD(LT) - XODB) 387,388,388
387 CONTINUE
XOD2 = XODB
CP02 = CP0(LT) + (CP0(LT+1) - CP0(LT))*(XODB - XOD(LT))
      / (XOD(LT+1) - XOD(LT))
1 CPA2 = CPA(LT) + (CPA(LT+1) - CPA(LT))*(XODB - XOD(LT))
      / (XOD(LT+1) - XOD(LT))
1 GO TO 389
388 CONTINUE
XOD2 = XOD(LT)
CP02 = CP0(LT)
CPA2 = CPA(LT)
LT = LT - 1
389 CONTINUE
XN1 = XOD1*DBAS
XN2 = XOD2*DBAS
X = XN1-XN2
CALL DIAM (XN1,D1)
CALL DIAM (XN2,D2)
CALL DLOD
FSUBZ = FSUBZ - FSBZ
AXLOAD = AXLOAD - AXLOD
BEND = BEND - BND - X*FSUBZ
IF (XOD2 - XODB) 386,386,445
386 CONTINUE
      C
      C
      C
      C
      C
      USING THE LOADS COMPUTED ABOVE, LINE LOADS WITH THE
      FACTOR OF SAFETY APPLIED ARE COMPUTED FOR BOTH THE WINDWARD
      AND LEEWARD SIDES OF THE BAY.
ANFIAX = C3*AXLOAD/(0.94248*DSUBB)
ANFIB = BEND*C3 / (0.23562* DSUBB **2)
ANFIMN = FS*(ANFIAX - ANFIB)
ANFIMX = FS*(ANFIAX + ANFIB)
RETURN

```

```

END
$IBFTC C53S7 DECK
SUBROUTINE DLOD
C INPUT VARIABLES--D1,D2,XOD1,XOD2,CPO1,CPO2,CPA1,CPA2,A3,A4,
C QB,DP
C OUTPUT VARIABLES--FSBZ,BND,AXLOD
COMMON /DLOAD/D1,D2,XOD1,XOD2,CPO1,CPO2,CPA1,CPA2,A3,A4,QB,DP,
1 FSBZ,BND,AXLOD
DX1 = XOD1 - XOD2
IF (DX1.LE..002)GO TO 10
DX2 = XOD1**2 -XOD2**2
DX3 = XOD1**3 - XOD2**3
DX4 = XOD1**4 -XOD2**4
B2 = (CPO1 - CPO2)/DX1
B1 = CPO1 - B2*XOD1
B4 = (CPO1 - CPO2)/(D1 - D2)
B3 = CPO1 - B4*D1
A2 = (CPA1 -CPA2)/DX1
A1 = CPA1 - A2*XOD1
AD2 = (D1-D2)/DX1
AD1 = D1 - AD2*XOD1
BTA1 = A1*AD1
BTA2 = 0.5*(A1*AD2 + A2*AD1)
BTA3 = A2*AD2/3.
FSBZ = A3*(BTA1*DX1 + BTA2*DX2 + BTA3*DX3)
BND = A4*(BTA1*DX2/2. - BTA1*XOD2*DX1 + BTA2*DX3/3.
1 -BTA2*DX1*XOD2**2 + BTA3*DX4/4. - BTA3*DX1*XOD2**3)
DD2 = (D1**2 - D2**2)/2.
DD3 = (D1**3 - D2**3)/3.
AXLOD = 1.5707963*(DD2*(QB*B3-DP) + DD3*B4*QB)
GO TO 20
10 DAV = 0.5*(D1 + D2)
CPOAV = 0.5*(CPO1 + CPO2)
FSBZ = A3*CPOAV*DX1*DAV
CPAAV = 0.5*(CPA1 + CPA2)
BND = A3*DX1*DX1*DAV*(CPA1/2.+2.*(CPA2-CPA1)/3.)
AXLOD = 1.5707963*(D1-D2)*DAV*(CPOAV* QB-DP)
20 CONTINUE
RETURN

```

LDS 194
DLN 001
DLN 002
DLN 003
DLN 004
DLN 005
DLN 006
DLN 007
DLN 008
DLN 009
DLN 010
DLN 011
DLN 012
DLN 013
DLN 014
DLN 015
DLN 016
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DLN 030
DLN 031
DLN 032
DLN 034
DLN 033
DLN 035
DLN 036
DLN 037
DLN 038


```

DLD 039
PRS 001
PRS 002
PRS 003
PRS 004
PRS 005
PRS 006
PRS 007
PRS 008
PRS 009
PRS 010
PRS 011
PRS 012
PRS 013
PRS 014
PRS 015
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PRS 028
PRS 029
PRS 030
PRS 031
PRS 032
PRS 033
PRS 034
PRS 035
PRS 036
PRS 037

END
$1BFTC C53S8 DECK
SUBROUTINE PRESUR
    THIS SUBROUTINE COMPUTES THE MAXIMUM DESIGN PRESSURE
    OCCURRING ANYWHERE ALONG THE LENGTH OF THE BAY ON BOTH THE
    WINDWARD AND LEEWARD SIDES OF THE BAY, USING EITHER THE
    INPUT PRESSURE DATA OR THAT COMPUTED IN SUBROUTINE AERO.
    COMMON/AERL0D/ALPHA, AMACH, NF
    COMMON /PRSLOD/ LTJNCT(10)
    COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
    COMMON/AERPRS/CPAA,CPOO,LPFL
    COMMON/PRESR/CPA(121),CPO(121),XOD(121),DELTAP, FS,LTMAX,SUMAL,
    1 QBAR,ALTOT
    COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
    COMMON /CHKLD/ C1,C3,C4,ALB,DELTAS,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
    1 CHKWND,CHKLEE,C2
    GO TO (399,400,417,415),LPFL
399 CONTINUE
    XODB = ALTOT/DBAS
    XOD1 = XODB
    QB = QBAR/144.
    LT1 = LTMAX + 1
    CPMX1 = CPO(LTMAX) + CPA(LTMAX)
    CPMN1 = CPO(LTMAX) - CPA(LTMAX)
    GO TO 415
400 CONTINUE
    L2 = LTMAX
    IF (NF.GT.1) L2 = LTJNCT(NF-1) - 1
    L1 = LTJNCT(NF)
    XODB = (ALTOT - SUMAL)/DBAS
    XOD1 = XODB
    IF (XOD1.GE.XOD(L2)) XOD1 = XOD(L2) - .00001
    DO 412 LT1 = L1,L2
    IF (XOD1.LT.XOD(LT1)) GO TO 414
412 CONTINUE
    414 LT = LT1
    CPOO = CPO(LT-1) + (CPO(LT) - CPO(LT-1))*(XOD1 - XOD(LT-1))
    1 / (XOD(LT) - XOD(LT-1))

```

```

CPAA      = CPA(LT-1) + (CPA(LT) - CPA(LT-1))*(XOD1 -XOD(LT-1))
1         /((XOD(LT) - XOD(LT-1)))
CPMX1     = CP00 + CPAA
CPMNI     = CP00 - CPAA
415 CONTINUE
LT        = LT1
CPMX      = CPMX1
CPMN      = CPMN1
ALN       = 0.
ALX       = 0.
417 CONTINUE
XOD2     = (ALTOT - SUMAL - ALB)/DBAS
IF (XOD2.LE.XOD(L1)) XOD2 = XOD(L1) + .00001
433 LT    = LT - 1
IF (LT.LE.L1) GO TO 419
IF (XOD(LT) - XOD2) 419,422,422
419 CONTINUE
LT        = LT + 1
CPAA      = CPA(LT-1) + (CPA(LT) - CPA(LT-1))*(XOD2 -XOD(LT-1))
1         /((XOD(LT) - XOD(LT-1)))
CP00     = CP0(LT-1) + (CP0(LT) - CP0(LT-1))*(XOD2 - XOD(LT-1))
1         /((XOD(LT) - XOD(LT-1)))
CPX      = CP00 + CPAA
CPN      = CP00 - CPAA
JFIN     = 0
GO TO 428
422 CONTINUE
CPN      = CP0(LT) - CPA(LT)
CPX      = CP0(LT) + CPA(LT)
JFIN     = 1
428 IF (CPMX - CPX) 424,425,425
424 CONTINUE
CPMX     = CPX
ALX      = (X0DB - XOD(LT))*DBAS
IF (JFIN.EQ.0) ALX=ALB
425 CONTINUE
IF (CPMN - CPN) 426,427,427
426 CONTINUE
CPMN     = CPN

```

PRS 038

PRS 039

PRS 040

PRS 041

PRS 042

PRS 043

PRS 044

PRS 045

PRS 046

PRS 047

PRS 048

PRS 049

PRS 050

PRS 051

PRS 052

PRS 053

PRS 054

PRS 055

PRS 056

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PRS 058

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PRS 066

PRS 067

PRS 068

PRS 069

PRS 070

PRS 071

PRS 072

PRS 073

PRS 074

PRS 075

PRS 076

```

ALN      = (XODB - XOD(LT))*DBAS
IF (JFIN.EQ.0) ALN=ALB
427 CONTINUE
IF (JFIN) 429,429,433
429 CONTINUE
PDESMX   = FS*(CPMX*QB - DELTAP)
PDESMN   = FS*(CPMN*QB - DELTAP)
RETURN
END
$IBFTC C53S9 DECK
SUBROUTINE INSTBL
C
C      THIS SUBROUTINE PERFORMS A GENERAL INSTABILITY ANALYSIS
C      FOR SANDWICH CYLINDERS. THE METHOD, DEVELOPED BY BO ALMROTH
C      OF LMSC SOLID MECHANICS LABORATORY, HANDLES AXIAL COMPRESSION
C      PLUS LATERAL PRESSURE. THE ANALYSIS IS ELASTIC, AND INCLUDES
C      TRANSVERSE SHEAR EFFECTS. A KNOCKDOWN FACTOR BASED ON (R/T)
C      EFFECTIVE IS APPLIED TO THE CLASSICAL LOAD. LATERAL PRESSURE
C      MAY BE EITHER BURSTING OR CRUSHING.
C      DIMENSION X(44),A2(2,44),Z(4)
COMMON/KLSS/XL
COMMON/INSTB/ E , G1, G2, XNU, T1, T2, RD, HT, CRG, K,P,KG
COMMON/KLASS/ CRW, XNCL, PBAR, XN, XN1, XN2, XN11, XN12, XN22
COMMON/INTRP/ X, A2, Z, XNMIN
COMMON/SFCTN/ GBAR, F
101 FORMAT(6E12.8)
201 FORMAT(1H0,49HCASE SKIPPED SINCE GBAR WAS OUTSIDE RANGE - GBAR=E11.4)
1.4)
202 FORMAT(1H0,43HCASE SKIPPED SINCE F WAS OUTSIDE RANGE - F=E11.4)
210 FORMAT(1H0,5HXNXR=E15.8)
KG      = 0
Z(1) = 1.E-02
Z(2)=1.E-03
Z(3)=1.E-04
Z(4)=1.E-05
E1=E
E2=E
TR=T1/T2
R=RD/HT

```

```

PRS 077
PRS 078
PRS 079
PRS 080
PRS 081
PRS 082
PRS 083
PRS 084
PRS 085
STB 001
STB 002
STB 003
STB 004
STB 005
STB 006
STB 007
STB 008
STB 009
STB 010
STB 011
STB 012
STB 013
STB 014
STB 015
STB 016
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STB 022
STB 023
STB 024
STB 025
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STB 027
STB 028
STB 029

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```

H=HT/T2
89 F=(TR**3+E2/E1)*(TR+1.0)**2/(H*H*TR*(1.0+E2/E1*TR))/12.0
GBAR=0.5*E1/G1*SQRT(E2/E1*TR/(1.0-XNU**2))/(R*H)*(1.0-(TR+1.0)/(2.0*H))
10*H)
IF(GBAR-X(1))52,10,10
52 XNMIN=0.123
GO TO 531
10 IF(GBAR-X(4))11,11,9
9 CONTINUE
KG = 1
GO TO 500
11 IF(F-Z(4))12,13,13
12 F = Z(4)
13 IF (F - Z(1)) 51, 51, 122
122 F = Z(1)
51 CALL INTERP
531 CALL CLASS
K=1
53 XNR=2.0*XNCL*E1*SQRT(E2/E1*TR/(1.0-XNU**2))/(H*R*R)
XNN=XNMIN/XNCL
C XMIN IS BASED ON G1 = G2 (SQUARECELL CORE). XMIN ALSO
C NEGLECTS PRESSURE EFFECTS. HOWEVER, THE KNOCKDOWN FACTOR IS
C APPROXIMATE, SO XMIN WILL BE ADEQUATE FOR OLTHER G1/G2
C VALUES AND FOR LOW PRESSURES.
IF(XNN-1.0)15,15,14
14 XNN=1.0
15 RTE=R*H*SQRT(TR+E2/E1)/SQRT(TR**3+E2/E1+12.0*H*H*TR*(1.0+E2/E1*TR)
1/((TR+1.0)**2))
IF (RTE .GT. 33.0) GO TO 17
16 PHI=1.0
GO TO 18
17 PHI=6.48/(RTE**0.54)
18 C=(PHI-0.12)/0.88
XNXR=XNR*(XNN+C*(1.0-XNN))
CRG=XNXR*RD
49 CONTINUE
500 CONTINUE
RETURN
END

```

STB 030
STB 031
STB 032
STB 033
STB 034
STB 035
STB 036
STB 037
STB 038
STB 039
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STB 065
STB 066
STB 067
STB 068

```

$IBFTC C53S10 DECK
SUBROUTINE INTERP
C      THIS SUBROUTINE IS UTILIZED BY INSTBL
      DIMENSION X(44),A2(2,44),Z(4)
      COMMON/INTRP/ X, A2, Z, XNMIN
      COMMON/SFCTN/ GBAR, F
      DIVDI(Q000FL,Q001FL,Q002FL,Q003FL,Q004FL,Q005FL,Q006FL)=Q001FL+(Q0TRP 001
106FL-Q000FL)/(Q002FL-Q000FL)*(Q003FL-Q001FL)+(Q006FL-Q000FL)*(Q006TRP 002
2FL-Q002FL)/(Q004FL-Q000FL)*(Q005FL-Q003FL)/(Q004FL-Q002FL)-(Q003FTRP 003
3L-Q001FL)/(Q002FL-Q000FL)
      ABSLOG(Q007FL,Q008FL,Q009FL,Q010FL,Q011FL)=Q008FL+(ALOG(Q011FL)-ALTRP 011
10G(Q007FL))/(ALOG(Q009FL)-ALOG(Q007FL))*(Q010FL-Q008FL)
      M=0
      N=0
      IT=-1
      DO 9 I=1,44
      IF(GBAR-X(I))8,10,9
          9 CONTINUE
10 M=I
      8 DO 11 J=1,4
      IF(F-Z(J))11,13,12
          11 CONTINUE
13 N=J
12 IF(GBAR-0.2)14,14,15
14 IF(F-Z(2))16,16,17
16 K=2
      GO TO 25
17 K=1
25 IF(M)18,18,19
19 XNMIN=A2(K,M)
      GO TO 24
18 IF(I-2)20,20,21
20 I=I+1
21 X0=X(I-2)
      X1=X(I-1)
      X2=X(I)
      XC=GBAR
27 IF(K-2)26,26,28
TRP 001
TRP 002
TRP 003
TRP 004
TRP 005
TRP 006
TRP 007
TRP 008
TRP 009
TRP 010
TRP 011
TRP 012
TRP 013
TRP 014
TRP 015
TRP 016
TRP 017
TRP 018
TRP 019
TRP 020
TRP 021
TRP 022
TRP 023
TRP 024
TRP 025
TRP 026
TRP 027
TRP 028
TRP 029
TRP 030
TRP 031
TRP 032
TRP 033
TRP 034
TRP 035
TRP 036
TRP 037
TRP 038

```

```

28 K=2
26 Y0=A2(K,I-2)
   Y1=A2(K,I-1)
   Y2=A2(K,I)
   XNMIN=DIVDI(X0,Y0,X1,Y1,X2,Y2,XC)
   IF(IT)24,29,30
24 IF(K-1)22,22,99
22 IF(N)23,23,99
23 IF(I-2)4,4,5
   4 I=I+1
   5 X0=X(I-2)
   X1=X(I-1)
   X2=X(I)
   XC=GBAR
   U0=Z(1)
   U1=Z(2)
   U#F
   Y0=A2(2,I-2)
   Y1=A2(2,I-1)
   Y2=A2(2,I)
   V1=DIVDI(X0,Y0,X1,Y1,X2,Y2,XC)
   V0=XNMIN
   XNMIN=ABSLOG(U0,V0,U1,V1,U)
   GO TO 99
15 IF(N)33,33,34
34 K=N
   GO TO 18
33 K=J-1
   IT=0
   GO TO 18
29 DO=XNMIN
   K=J
   IT=1
   GO TO 18
30 D1=XNMIN
   B0=Z(J-1)
   B1=Z(J)
   IF(D0-D1)32,99,32
32 XNMIN=ABSLOG(B0,D0,B1,D1,F)
TRP 039
TRP 040
TRP 041
TRP 042
TRP 043
TRP 044
TRP 045
TRP 046
TRP 047
TRP 048
TRP 049
TRP 050
TRP 051
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TRP 053
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TRP 065
TRP 066
TRP 067
TRP 068
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TRP 070
TRP 071
TRP 072
TRP 073
TRP 074
TRP 075
TRP 076
TRP 077

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```

99 RETURN
END
$IBFTC C53S11 DECK
SUBROUTINE CLASS
NON-SYMMETRIC BUCKLING PROGRAM. F. BROGAN. B. ALMROTH, B. BURNS
WHEN K=0, SUBROUTINE CLASS PICKS INITIAL ESTIMATES FOR THE
INDEPENDENT VARIABLES.
IF K GREATER THAN 0, PREVIOUS SOLUTIONS ARE USED FOR STARTING
ESTIMATES FOR THE NEXT CASE.
IF IOUT=0, NO INTERMEDIATE OUTPUT WILL BE PRINTED.
SET IOUT=1 IN SUBROUTINE CLASS IF INTERMEDIATE OUTPUT IS DESIRED.
COMMON/KLSS/XL
COMMON/KLASS/ CRW, XNCL, PBAR, XN, XN1, XN2, XN11, XN12, XN22
COMMON/INSTB/ EX, G1, G2, ZNU, T1, T2, RD, HT, CRG, K ,P
COMMON/SFCTN/ GBAR, F
901 FORMAT ( 48H0 THE VALUES OF N, N1, N2, Z1, Z2, B, AND E ARE /
1 7E16.8 )
902 FORMAT ( 59H0 THE NON-SYMMETRIC BUCKLING PROGRAM HAS NOT CONVERGED
1 IN 14, 12H ITERATIONS. / 3H B= E16.8, 2HE= E16.8 )
903 FORMAT ( 48H0 THE NON-SYMMETRIC BUCKLING HAS CONVERGED IN 14,
1 11H ITERATIONS / 33H THE VALUES OF N, B, AND E ARE / 3E16.8)
P12=3.1415927**2
ZN1=SQRT (1.-ZNU**2)
AP=P12*ZN1*RD*HT/XL**2
KBR=AMAX1(KBR,1)
IBR=KBR
CFT=.75
IOUT=0
DEL=.1
DLTA=.0001
EH=EX*HT
ZMU=G1/G2
DELX=DEL
G=GBAR
M1=75
M2=75
M4=40
PBAR=P/EH*RD**2*SQRT ((1.-ZNU**2)/(T1*T2))

```

TRP 078
TRP 079
CLS 001
CLS 002
CLS 003
CLS 004
CLS 005
CLS 006
CLS 007
CLS 008
CLS 009
CLS 010
CLS 011
CLS 012
CLS 013
CLS 014
CLS 015
CLS 016
CLS 017
CLS 018
CLS 019
CLS 020
CLS 021
CLS 022
CLS 023
CLS 024
CLS 025
CLS 026
CLS 027
CLS 028
CLS 029
CLS 030
CLS 031
CLS 032
CLS 033
CLS 034
CLS 035
CLS 036

```

PS=PBAR
KM=K
ITER=0
IF (K) 101, 101, 104
101 IF (G-.5) 102, 103, 103
102 ZI=1./((2.-4.*G)
GO TO 104
103 ZI=1./((2.*SQRT (F))
104 CONTINUE
DX=.001*ZI
DX2=2.*DX
DXX=DX*DX
105 CONTINUE
F1=SFUN(ZI-DX)
F2=SFUN(ZI)
F3=SFUN(ZI+DX)
DZ=((F3-F1)/DX2*DXX)/(F3-2.*F2+F1)
ZI=ZI-DZ
IF (ABS (DZ/ZI)-DLTA) 120, 120, 110
110 ITER=ITER+1
IF (ITER-M1) 105, 105, 130
120 AN=F2
IF (IOUT) 150, 150, 121
121 WRITE (6,908)ITER, F2, ZI
GO TO 150
130 WRITE (6,906)ITER
AN=1.
906 FORMAT (40H0 THE SYMMETRIC BUCKLING HAS FAILED IN 14,
1 12H ITERATIONS )
908 FORMAT (10H0 ITER= 14, 5H AN= E16.7, 4H Z= E16.8)
150 CONTINUE
ITER=0
IT2=0
B=BS
E=ES
IF (K) 3, 3, 4
3 B=1.
E=1.
KBR=1

```

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CLS 037
CLS 038
CLS 039
CLS 040
CLS 041
CLS 042
CLS 043
CLS 044
CLS 045
CLS 046
CLS 047
CLS 048
CLS 049
CLS 050
CLS 051
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CLS 070
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CLS 072
CLS 073
CLS 074
CLS 075

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CLS 076
 CLS 077
 CLS 078
 CLS 079
 CLS 080
 CLS 081
 CLS 082
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 CLS 099
 CLS 100
 CLS 101
 CLS 102
 CLS 103
 CLS 104
 CLS 105
 CLS 106
 CLS 107
 CLS 108
 CLS 109
 CLS 110
 CLS 111
 CLS 112
 CLS 113

IBR=1
 Z2=0.
 PBAR=0.
 BS=B
 ES=E
 IF (AP-1.) 4, 4, 7
 7 E=2.*AP
 ES=E
 4 CONTINUE
 Y=1.-ZNU
 Y1=Y+1.
 Y2= (1.+F)/4.
 Y3=.25*G
 Y4=2.*Y*ZMU
 Y5=2.*ZMU*ZNU
 A=1./ (G*E)
 A1=A+Y
 BB=B*B
 V=1.+BB
 W=2.*B
 V1=Y*ZMU*BB
 V2=A+V1
 W2=Y*ZMU*W
 V3=A+Y+BB
 W3=W
 V4=V2+ZMU
 W4=W2
 V5=V3*V4-ZMU*BB
 W5=W3*V4+V3*W4-ZMU*W
 V11=1./BB
 W11=-W*V11**2
 V14=1./V5
 W14=-W5*V14**2
 V6=V/V5
 W6=V*W14+W*V14
 V7=V2*V6
 W7=W2*V6+V2*W6
 V9=V*V/BB

W9=W+W11
 V15=1./V9
 W15=-W9*V15**2
 B2=-B*V7
 B2B=-V7-B*W7
 C2=-A1*V6
 C2B=-A1*W6
 V12=Y3*(B2**2+ZMU*C2**2)
 W12=2.*Y3*(B2*B2B+ZMU*C2*C2B)
 V20=A+4.*BB
 W20=8.*B
 V21=1./V2
 V23=V21**2
 W21=-W20*V23
 V22=8.*B*BB
 W22=24.*BB
 B1=-V21*V22
 B12=-V22*W21-W22/V20
 V24=ZMU*C2
 W24=ZMU*C2B
 B1B=B1**2
 V25=BB*B1B
 V26=B1*B12
 W25=2.*(B*B1B+BB*V26)
 V30=B2+B*V24
 W30=B2B+V24+B*W24
 V31=B*(Y1+BB)
 W31=Y1+3.*BB
 V32=B2*V31
 W32=B2*W31+B2B*V31
 V33=ZMU*(1.+Y1*BB)
 W33=Y1*ZMU*W
 V34=C2*V33
 W34=C2*W33+C2B*V33
 V35=2.*Y*V30**2
 W35=4.*Y*W30*V30
 V36=B2*C2
 W36=B2*C2B+B2B*C2
 V37=Y5*BB*V36

CLS 114
 CLS 115
 CLS 116
 CLS 117
 CLS 118
 CLS 119
 CLS 120
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 CLS 149
 CLS 150
 CLS 151
 CLS 152

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W37=Y5*(V36+B*W36)
S2=2.*(V25+V24**2+V37+V35)+4.*(V32+V34)
S2B=2.*(W25+2.*V24*W24+W37+W35)+4.*(W32+W34)
V16=V12+E*S2/8.-PBAR
W16=W12+E*S2B/8.
AE=-G*A*A
U2=AE
U5=AE*(V3+V4)
U14=-U5*V14**2
U6=V*U14
U7=U2*V6+V2*U6
B2E=-B*U7
U21=-AE*V23
B1E=-V22*U21
C2E=-A1*U6-AE*V6
U24= ZMU*C2E
U12=2.*Y3*(B2*B2E+V24*C2E)
U25=2.*BB*B1*B1E
U30=B2E+B*U24
U32=B2E*V31
U34=C2E*V33
U35=4.*Y*V30*U30
U36= B2*C2E+B2E*C2
U37=Y5*B*U36
S2E=2.*(U25+2.*V24*U24+U37+U35)+4.*(U32+U34)
U16=U12+S2/8.+E*S2E/8.
XN=Y2*E*V9+V15/E+V11*V16
EB=AP*W11
XN1=Y2*E*W9+W15/E+V11*W16+W11*V16
XN2=Y2*V9+G*V15*AE+V11*U16
FN1=XN1+XN2*EB
IF (KM) 34, 34, 48
34 IF (ITER) 40, 40, 35
35 IF (XN-XNA) 40, 45, 45
40 CONTINUE
XNAB=SQRT (XN1**2+XN2**2)/DELX
XNA=XN
IT2=IT2+1

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CLS 153
CLS 154
CLS 155
CLS 156
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CLS 188
CLS 189
CLS 190

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GO TO (36, 37), IBR
36 CONTINUE
  Z1=XN1/XNAB
  Z2=XN2/XNAB
  GO TO 39
37 CONTINUE
  Z1=DELX*FN1/ABS (FN1)
  Z2=0.
39 CONTINUE
  IF (IT2-M4) 47, 47, 44
44 Z1=CFT*Z1
  Z2=CFT*Z2
  DELX=CFT*DELX
  M4=15
  IT2=0
  GO TO 47
45 IF (DELX-.1*DLTA) 60, 60, 46
46 DELX=DELX/3.
  B=B+.66667*Z1
  E=E+.66667*Z2
  Z1=Z1/3.
  Z2=Z2/3.
  M4=10
  IT2=0
  GO TO 49
48 CONTINUE
  WB2=Y4
  WB5=2.*(W3*W4+V4+Y*ZMU*V3-ZMU)
  WB11=6.*V11**2
  WB14=-2.*W5*V14*W14-WB5*V14**2
  WB6=2.*(W*W14+V14)+V*WB14
  WB7=2.*W2*W6+WB2*V6+V2*WB6
  WB9=2.+WB11
  WB15=-WB9*V15**2+2.*W9**2*V15**3
  B2BB=-2.*W7-B*WB7
  C2BB=-A1*WB6
  WB12=2.*Y3*(B2*B2BB+B2B**2+ZMU*(C2*C2BB+C2B**2))
  WB21=-2.*W20*V21*W21-8.*V23
  B12B=-2.*(W22*W21)-V22*WB21-48.*B/V20

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CLS 191
CLS 192
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CLS 226
CLS 227
CLS 228
CLS 229

```

WB24=ZMU*C2BB
 WB25=2.*(B1B+4.*V26*B+88*(B12**2+B1*B12B))
 WB30=B2BB+2.*W24+ZMU*B*C2BB
 WB31=6.*B
 WB32=2.*B2B*W31+B2*WB31+B2BB*V31
 WB34=2.*C2B*W33+C2*2.*Y1*ZMU+C2BB*V33
 WB35=4.*Y*(W30**2+WB30*V30)
 WB36=2.*B2B*C2B+B2*C2BB+B2BB*C2
 WB37=Y5*(2.*W36+B*WB36)
 S2BB=2.*(WB25+2.*W24**2+2.*V24*WB24+WB37+WB35)+4.*(WB32+WB34)
 WB16=WB12+E*S2BB/8.
 AEE=-2.*G*AE*A
 USE=2.*AE**2+AEE*(V3+V4)
 USB=AE*(W3+W4)
 U14E=-2.*U5*V14*U14-USE*V14**2
 W14E=-U5B*V14**2-2.*U5*V14*W14
 U6E=V*U14E
 U6B=V*W14E+W*U14
 U7E=2.*U2*U6+AEE*V6+V2*U6E
 U7B=U2*W6+W2*U6+V2*U6B
 B2EE=-B*U7E
 B2EB=-U7-B*U7B
 U21E=-2.*AE*V21*U21-AEE*V23
 U21B=-2.*AE*V21*W21
 B1EE=-V22*U21E
 B1EB=-W22*U21-V22*U21B
 C2EE=-A1*U6E-AEE*V6-2.*AE*U6
 C2EB=-A1*U6B-AE*W6
 U24E=ZMU*C2EE
 U24B=ZMU*C2EB
 U12E=2.*Y3*(B2E**2+B2*B2EE+U24*C2E+V24*C2EE)
 U12B=2.*Y3*(B2*B2EB+B2B*B2E+W24*C2E+V24*C2EB)
 U25E=2.*BB*(B1E**2+B1*B1EE)
 U25B=4.*B*B1*B1E+2.*BB*(B12*B1E+B1*B1EB)
 U30E=B2EE+B*U24E
 U30B=B2EB+U24+B*U24B
 U32E=B2EE*V31
 U32B=B2EB*V31+B2E*W31

CLS 230
 CLS 231
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 CLS 267

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U34E=C2EE*V33
U34B=C2E*W33+C2EB*V33
U35E=4.*Y*(U30**2+V30*U30E)
U35B=4.*Y*(V30*U30B+W30*U30)
U36E=2.*B2E*C2E+B2*C2EE+B2EE*C2
U36B=B2B*C2E+B2*C2EB+B2EB*C2+B2E*C2B
U37E=Y5*B*U36E
U37B=Y5*U36+Y5*B*U36B
S2EE=2.*(U25E+2.*(U24**2+V24*U24E)+U37E+U35E)+4.*(U32E+U34E)
S2EB=2.*(U25B+2.*(W24*U24+V24*U24B)+U37B+U35B)+4.*(U32B+U34B)
U16E=U12E+S2E/4.+E*S2EE/8.
U16B=U12B+S2B/8.+E*S2EB/8.
EBB=AP*WB11
EB2=EB**2
XN11=Y2*E*WB9+WB15/E+2.*W11*W16+V11*WB16+WB11*V16
XN12=Y2*W9+G*W15*AE+V11*U16B+W11*U16
XN22=G*V15*AE+ V11*U16E
GO TO (41, 42), IBR
41 CONTINUE
XK=XN12/XN11
Z2=(XN2-XK*XN1)/(XN22-XK*XN12)
Z1=(XN1-XN12*Z2)/XN11
GO TO 47
42 CONTINUE
FN11=XN11+2.*XN12*EB+XN22*EB2+XN2*EBB
Z1=FN1/FN11
Z2=0.
47 CONTINUE
B=B-Z1
E=E-Z2
49 GO TO (55, 57), IBR
55 IF (E-AP/B**2) 56, 56, 58
56 IBR=2
KBR=2
57 E=AP/B**2
58 CONTINUE
D1=ABS (Z1/B)
D2=ABS (Z2/E)
IF (IOUT) 53, 53, 51

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CLS 268
CLS 269
CLS 270
CLS 271
CLS 272
CLS 273
CLS 274
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CLS 305
CLS 306

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51 CONTINUE
   WRITE (6,901) XN, XN1, XN2, Z1, Z2, B, E
53 CONTINUE
   IF (D1-DLTA) 50, 50, 52
50 IF ( D2-DLTA) 60, 60, 52
52 ITER=ITER+1
   IF (ITER-M2) 4, 4, 96
96 IF (KM) 97, 97, 98
97 KM=1
   PBAR=PS
   ITER=0
   GO TO 4
60 WRITE (6,903) ITER, XN, B, E
   IF (KM) 97, 97, 68
68 GO TO (61, 62), IBR
61 IBR=2
   ES=E
   BS=B
   XNS=XN
   B=1.
   ITER=0
   GO TO 7
62 GO TO (63, 65), KBR
63 IF (XN-XNS) 66, 64, 64
64 XN=XNS
   GO TO 66
65 CONTINUE
   BS=B
   ES=E
66 CONTINUE
   XNCL=AMIN1 (AN,XN)
   XNCL=MIN1 (AN,XN)
   IF (KM) 97, 97, 99
98 WRITE (6, 902) ITER, B, E
   XNCL=AN
99 RETURN
   END
$IBFTC C53S12 DECK

```

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CLS 307
CLS 308
CLS 309
CLS 310
CLS 311
CLS 312
CLS 313
CLS 314
CLS 315
CLS 316
CLS 317
CLS 318
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CLS 320
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CLS 336
CLS 337
CLS 338
CLS 339
CLS 340
CLS 341
CLS 342
CLS 343
SFN 001

```

```

FUNCTION SFUN (Z)
DIMENSION X(44),A2(2,44),Z(4)
COMMON/SFCTN/ GBAR, F
ZX=4.*Z
SFUN=(1.+F)*Z+1./ZX-ZX*Z/(1./GBAR+ZX)
RETURN
END
$IBFTC C53S13 DECK
SUBROUTINE IREQ
COMMON/CONFG/THTA(10),ALF(10),NFMAX,DBAS,DMN(10),ICONFG
COMMON/AERLOD/ALPHA,AMACH,NF
COMMON /CHKLD/ C1,C3,C4,ALB,DELTA,CHK,RAXMIN,RAXMAX,RPMIN,RPMAX,
1  CHKWND,CHKLEE,C2
COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESMX,E
COMMON/IRG/ TTM, PI, ALCONE, C6, AIREG
IF (THTA(NF).LT. 0.01) GO TO 990
AIREG = C6*ALB *((DSUBB - C2*ALB )**2)*(PDESMX**1.33333)
1 /TTM**0.33333
GO TO 994
C
C DETERMINE RING MOMENT OF INERTIA REQUIREMENTS TO PREVENT
C GENERAL INSTABILITY BY THE METHOD OF SHANLEY. SEE BECKER'S
C HANDBOOK OF STRUCTURAL STABILITY, VOL. 6, P24. (NACA TN 3786)
990 AIREG = (ALB * (2.5E - 04) * PI * ANFIMX * (DSUBB / 2.) ** 4)/
1 (E * ALCONE)
994 CONTINUE
RETURN
END
$IBFTC C53S14 DECK
SUBROUTINE RING
COMMON/RNG/ IKIND, AOT, BOT, COT, TSTD(30), K, AIRING, DD
COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
RINGS ARE ASSUMED TO BE FORMED SHEET METAL OF STANDARD
GAUGE. IKIND INDICATES THE TYPE OF RING...1 = ANGLE, 2 = ZEE,
3 = HAT. BASE FLANGE WIDTH IS GIVEN BY B, RING DEPTH IS A,
AND OUTSTANDING OR UPPER FLANGE WIDTH IS GIVEN BY C. SHEET
THICKNESS IS INDICATED BY TT.
TT = TSTD(K)
A = AOT * TT
B = BOT * TT

```

SFN 002
SFN 003
SFN 004
SFN 005
SFN 006
SFN 007
SFN 008
IRQ 001
IRQ 002
IRQ 003
IRQ 004
IRQ 005
IRQ 006
IRQ 007
IRQ 008
IRQ 009
IRQ 010
IRQ 011
IRQ 012
IRQ 013
IRQ 014
IRQ 015
IRQ 016
IRQ 017
IRQ 018
IRQ 019
IRQ 020
RNG 001
RNG 002
RNG 003
RNG 004
RNG 005
RNG 006
RNG 007
RNG 008
RNG 009
RNG 010
RNG 011
RNG 012


```

C
GO TO (1020,1030,1040), IKIND
1020 AST = B*TT + (A-TT) * TT
DD = ( A*A*TT/2. + (B-TT)*(TT/2.)*2)/AST
AIB = ((B-TT) * TT**3)/12.
AIF = (A**3)*TT/12.
AB = (B-TT)*TT
AF = A*TT
DB = DD - TT/2.
DF = A/2. -DD
AIST=AIB+AIF+AB*DB*DB+AF*DF*DF
GO TO 105
1030 AST = A*TT + 2.*(B-TT)*TT
DD = A/2.
AIB = (B-TT) * (TT**3)/12.
AIW = TT*(A**3)/12.
DB = (A-TT)/2.
AB = (B-TT)*TT
AIST=AIB*2.+AIW+AB*DB*DB*2.
GO TO 105
1040 AST=2.*B*TT +C*TT + 2.*(A-2.*TT)*TT
DD = ((B-TT)*TT*TT + A*A*TT + (C-2.*TT)*(A-TT/2.)*TT)/AST
AB = 2. * (B-TT)*TT
AW = 2.0 * A * TT
AT = (C -2.*TT)*TT
AIB = (B-TT)*(TT**3)/12.
AIW = TT*(A**3)/12.
AIT = (C-2.*TT)*(TT**3)/12.
DB = DD-TT/2.
DT = (A-TT/2. -DD)
DW = DD - A/2.
AIST=AIB*2.+AIW*2.+AIT+AB*2.*DB*DB
1+AW*DW*DW*2.+AT*DT*DT
1050 CONTINUE
ECC = DD + .5 *TF
Z = (AST * ECC) / (AST + B * TF)
CALCULATE MOMENT OF INERTIA OF RING,INCLUDING ONE FACE.
AISTT = AIST + (B * TF**3 / 12.) + AST * (ECC - Z)**2 +
RNG 013
RNG 014
RNG 015
RNG 016
RNG 017
RNG 018
RNG 019
RNG 020
RNG 021
RNG 022
RNG 023
RNG 024
RNG 025
RNG 026
RNG 027
RNG 028
RNG 029
RNG 030
RNG 031
RNG 032
RNG 033
RNG 034
RNG 035
RNG 036
RNG 037
RNG 038
RNG 039
RNG 040
RNG 041
RNG 042
RNG 043
RNG 044
RNG 045
RNG 046
RNG 047
RNG 048
RNG 049
RNG 050

```

```

1 B * TF * Z**2
AA = A
AIRING = AIST
RETURN
END
$IBFTC C53S15 DECK
BLOCK DATA
DIMENSION X(44),A2(2,44),Z(4)
COMMON/RNG/ IKIND, AOT, BOT, COT, TSTD(30), K, AIRING, AST, DD
COMMON/INTRP/ X, A2, Z, XNMIN
DATA TSTD/ .008, .012, .016, .020, .025, .032, .040, .050, .063,
1 .071, .080, .090, .100, .125, .190, .250, .312, .375, .438, .500,
2 .562, .625, .688, .750, .812, .875, .938, 1.000, 1.125, 1.250 /
DATA X/.003, .004, .005, .006, .007, .008, .009, .01, .02, .03, .04, .05, .06,
1 .07, .08, .09, .1, .2, .3, .4, .5, .6, .7, .8, .9, 1.1, 1.2, 1.3, 1.4, 1.5, 1.6, 1.7, 1.8,
2 1.9, 1.10, 1.20, 1.30, 1.40, 1.50, 1.60, 1.70, 1.80, 1.90, 1.100,
END
$IBFTC C53S16 DECK
SUBROUTINE RSTRES
COMMON ANFIMN, ANFIMX, DSUBB, PDESMN, PDESMX, E
COMMON /CHKLD/ C1, C3, C4, ALB, DELTAS, CHK, RAXMIN, RAXMAX, RPMIN, RMAX,
1 CHKWND, CHKLEE, C2
COMMON /RSTRSS/ AA, TF, Z, CTH, AST, AISTT, FRING
ZFLG = AA + .5 * TF - Z
ANTHTA = PDESMX * DSUBB / (2. * CTH)
ANRING = ANHTA * ALB * (AST / (AST + 2. * ALB * TF))
C ESTIMATE COMPRESSIVE RING STRESS DUE TO OUT-OF-ROUNDNESS
C AND ASSYMMETRY OF LOADING. (F = PR)
FBEND = ANRING * ZFLG / AISTT
FRING = ANRING / AST + FBEND
RETURN
END
$IBFTC C53S17 DECK
SUBROUTINE TNSST
C THE FOLLOWING SUBROUTINE CALCULATES NOSE CAP SKIN THICK-
C NESS REQUIRED FOR STRUCTURAL INTEGRITY. THE FAILURE CRITERION
C USED IS THAT PRESENTED FOR NON-SHALLOW SPHERICAL CAPS IN THE
C LMSC STRUCTURAL METHODS HANDBOOK, SECTION 6.32.1, DATED
C 30 SEPTEMBER 1962.
RNG 051
RNG 052
RNG 053
RNG 054
RNG 055
BLK 001
BLK 002
BLK 003
BLK 004
BLK 005
BLK 006
BLK 007
BLK 008
BLK 009
BLK 010
BLK 011
BLK 012
RSS 001
RSS 002
RSS 003
RSS 004
RSS 005
RSS 006
RSS 007
RSS 008
RSS 009
RSS 010
RSS 011
RSS 012
RSS 013
RSS 014
RSS 015
TNS 001
TNS 002
TNS 003
TNS 004
TNS 005
TNS 006
TNS 007

```

```

COMMON/TNOS/TMINN,PDSPH,TCAPST,RCAP
COMMON ANFIMN,ANFIMX,DSUBB,PDESMN,PDESXPR
C          CONSTANT USED IN CALCULATING THE COLLAPSE PRESSURE FOR
C          THE NOSE CAP IS
46 A      =0.606*E/(RCAP**2)
C          SHELL THICKNESS IS SET EQUAL TO TMINN, AND COLLAPSE
C          PRESSURE OF THE NOSE CAP IS CALCULATED. IF COLLAPSE PRESSURE
C          IS LESS THAN THE DESIGN PRESSURE, SKIN THICKNESS IS INCREASED
C          BY INCREMENTS OF 0.001 INCH UNTIL COLLAPSE PRESSURE IS EQUAL
C          TO OR GREATER THAN THE DESIGN PRESSURE.
          TCAPST = TMINN -0.001
90 TCAPST = TCAPST + 0.001
B      = 0.04*SQRT (RCAP/TCAPST)
PCOLL = A * (TCAPST ** 2)/EXP(B)
IF (PCOLL - PDSPH) 90,91,91
91 CONTINUE
RETURN
END
TNS 008
TNS 009
TNS 010
TNS 011
TNS 012
TNS 013
TNS 014
TNS 015
TNS 016
TNS 017
TNS 018
TNS 019
TNS 020
TNS 021
TNS 022
TNS 023
TNS 024
TNS 025

```

APPENDIX B
DEFINITIONS OF VARIABLE NAMES

A	-	The length of the free side of a honeycomb cell, in.
AA	-	The height of the stiffening ring, in.
AFACT	-	Used to determine which equations to use in determining wrinkling stress.
AGWRIN	-	Allowable face wrinkling stress, psi.
AIREQ	-	Moment of inertia required of the stiffening ring cross-section, in. ⁴
AIRING	-	Moment of inertia of the stiffening ring, in. ⁴
AISTT	-	Moment of inertia of ring and effective skin, in. ⁴
ALB	-	Bay length, in.
ALCAP	-	Axial length of nose cap, in.
ALCONE	-	Frustum length, in.
ALF (NF)	-	Length of frustum number NF, in.
ALMX2	-	Axial distance from base of bay to top of frustum, in.
ALPHA	-	Angle of attack, degrees when read in and radians when used in computations.
ALTOT	-	Total length of fairing, in.
AMACH	-	Mach number.
AMU	-	Poisson's ratio.
ANFLAX	-	Line load contributed by axial loading on bay, lbs/in.
ANFIB	-	Line load contributed by bending moments on bay, lbs/in.
ANFICR	-	Critical line load, lbs/in.
ANFIMN	-	Total line load on windward side of fairing multiplied by factor of safety, lbs/in.
ANFIMX	-	Total line load on leeward side of fairing multiplied by factor of safety, lbs/in.
AOT	-	The ratio A/t (see Figure 3).

APRC	-	An equivalent radius of curvature of the particular bay under consideration.
AST	-	Cross-sectional area of ring, sq. in.
AXLDCP	-	Axial load contributed by nose cap, lbs.
AXLOAD	-	Axial load at some specified location on fairing, lbs.
B	-	The length of the bonded side of a honeycomb cell, in.
BEND	-	Bending moment at some specified location on fairing, lb-in.
BETA	-	The acute angle formed by the sides of a honeycomb cell, radians.
BETADG	-	The acute angle formed by the sides of a honeycomb cell, degrees.
BNDCAP	-	Bending moment of the nose cap about its base, lbs/in.
BOT	-	The ratio of width of attaching ring flange to ring thickness (see Figure 3).
CASE	-	The number of a particular set of data in a sequence of runs.
CBETA	-	Cosine of the angle BETA.
CCA	-	Honeycomb core allowable compressive stress, psi.
CDCAP	-	Drag coefficient for the spherical nose cap with the base area of the nose cap as a reference area.
CELLWD	-	Calculated honeycomb cell size, in.
CELSIZ	-	Honeycomb cell size reduced to a standard size, in.
CNCAP	-	Normal force coefficient per radian angle of attack for the spherical nose cap using nose cap base area as a reference area, /radian.
COT	-	The ratio of the width of the cap of stiffener to ring thickness (see Figure 3).
CPA(LT)	-	The change in pressure coefficient on either the windward or leeward side of the fairing due to angle of attack at station LT. (See Figures 4 and 5.)
CPAA	-	Same as CPA(LT).

CPO(LT)	-	Pressure coefficient at zero angle of attack at station LT.
CPOO	-	Same as CPO(LT).
CRG	-	Critical line load for general panel instability, lbs/in.
CSA	-	Sandwich core allowable shear stress, psi.
CTA	-	Sandwich core allowable tensile stress, psi.
D (I)	-	Small diameter of the Ith bay, in.
DBAS	-	Base diameter of fairing, in.
DBASE	-	Base diameter of a frustum, in.
DBNSD	-	Inside base diameter of bay, in.
DD	-	Distance from the centroid of the stiffening ring to the inside bay wall, in.
DELTAP	-	Difference between internal and free-stream pressure, psi.
DELTAS	-	Bay skin thickness, in.
DINSD	-	Inside upper diameter of bay, in.
DMIN	-	Small diameter of frustum, in.
DMN	-	Same as DMIN.
DOD(LT)	-	Ratio of local diameter to fairing base diameter at station LT.
DOVDB(NF)	-	Ratio of small diameter of frustum NF to base diameter of fairing.
DSUBB	-	Base diameter of bay, in.
DUSE1	-	Diameter of area useful for payload at the base of the bay. (See Line SWD 0507 of program listing in Appendix A.)
DUSE2	-	Diameter of area useful for payload at top of bay. (See Line SWD 0508 of program listing in Appendix A.)
E	-	Modulus of elasticity of structural material, in.
ECORE	-	Modulus of elasticity of core material, psi.
EFACE	-	Modulus of elasticity of face material, psi.

ELMIN(NF)	-	Minimum bay length for frustum NF, in.
ENFIMN (I)	-	ANFIMN for Ith bay, lbs/in.
ENFIMX (I)	-	ANFIMX for Ith bay, lbs/in.
ENSUBR	-	Radius conversion factor for conical frustums.
FCA	-	Sandwich face allowable compressive stress, psi.
FCFB	-	Ring outstanding flange allowable buckling stress, psi.
FFAX (I)	-	Face axial stress at base of Ith bay, psi.
FFB (I)	-	Face bursting stress in Ith bay, psi.
FFC (I)	-	Face compressive stress in Ith bay, psi.
FFDMP	-	Face dimpling stress, psi.
FFS (I)	-	Face shear stress due to aerodynamic lift, psi.
FFWR (I)	-	Face wrinkling stress, psi.
FRING	-	Maximum compressive stress in ring flange, psi.
FS	-	Factor of safety.
FSA	-	Sandwich face allowable shear stress, psi.
FSBZCP	-	Shear force contributed by nose cap, lbs.
FSUBZ	-	Shear force at a specified location on the fairing, lbs.
FTA	-	Sandwich face allowable tensile stress, psi.
FZ (I)	-	Shear force at the base of the Ith bay, lbs.
G	-	Modulus of shear, psi.
GCORE	-	Modulus of shear of core material, psi.
GC1	-	Effective shear modulus of the honeycomb cellular material in the ribbon direction, psi.
GC2	-	Effective shear modulus of the core in the direction perpendicular to the ribbons, psi.
GFACE	-	Modulus of shear of face material, psi.
G1	-	Effective shear modulus of the core material corrected for height to cell size ratio, in the ribbon direction, psi.

- G2 - Effective shear modulus corrected for height to cell size ratio, perpendicular to the ribbon direction, psi.
- H - Distance between midplanes of facing sheets, in.
- HUSE - Useful axial length of nose cap, in.
- I - Index indicating bay. Numbering begins at base of fairing.
- IKIND - Type of ring. 1 = angle, 2 = Zee, 3 = hat.
- IMAX - Total number of bays in fairing.
- IMX(NF) - Number of bays from bottom of fairing to top of frustum NF.
- IW - Index indicating bay number within a frustum.
- I1 - Lower index used for writing output data.
- I2 - Upper index used for writing output data.
- J - Index indicating parameter associated with face thickness T (J).
- JF - Index of skin thickness which is optimum for a bay.
- JG - Index used for determining minimum adequate ribbon thickness.
- KEY - Input parameter indicating type of output desired. (See Section 3.0 of the TECHNICAL DISCUSSION.)
- KG - Parameter generated by Subroutine INSTBL, indicating that GBAR has exceeded the range in which it is valid.
- KTFSTD - An input parameter which tells if standard gauges are to be used for sandwich faces.
- RTRSTD - An input parameter which tells if standard gauges are to be used for core ribbons.
- KY - An optional control used to obtain faster convergence in Subroutine INSTBL.

- LPFL - An integer used to control computation of lateral pressures.
- LPRES - An integer indicating type of pressure profile to be read in. (See Section 3.0 of TECHNICAL DISCUSSION.)
- LSTOP - An integer indicating that the last pressure profile data card has been read.
- LT - Index that identifies a particular point on the fairing.
- LTMAX - Total number of pressure profile data points.
- MAT - An integer indicating the material to be used. (See Section 3.0 of the TECHNICAL DISCUSSION.)
- MATF - An integer indicating the type of material to be used for sandwich faces.
- NBAY - Number of bays in a frustum.
- NF - Index indicating frustum number.
- NFMAX - Total number of frustums in the fairing.
- P - Pressure differential, equal to either PDESMN or PDESMX, as appropriate, psi.
- PDESMN - Maximum pressure differential across the sandwich, multiplied by the factor of safety, on the leeward side of the bay, psi.
- PDESMX - Maximum pressure differential across the sandwich, multiplied by the factor of safety, on the windward side of the fairing, psi.
- PDIFF - Maximum burst pressure differential, psi.
- PDSPH - Maximum pressure differential across the nose cap skin, multiplied by a factor of safety, psi.
- PHI - Angle between the applied load and the direction of the honeycomb ribbon.

PSTAT	-	Free-stream pressure, psi.
QBAR	-	Dynamic pressure, lbs/sq. ft.
RCAP	-	Radius of spherical nose cap, in.
RHO	-	Material density, lbs./cu. in.
ROCORE	-	Core material density, lbs/cu. in.
RUSE	-	Radius of nose cap volume which is useful for payload, in.
SCAP	-	Surface area of spherical nose cap, sq. in.
SLOPT	-	Sum of bay lengths within a frustum, in.
SUMAL	-	Distance from the base of the fairing to the base of the bay, in.
TCAP	-	Thickness of nose cap skin, in.
TCAPST	-	Thickness of nose cap skin required to withstand pressure loads, in.
TCAPTH	-	Thickness of nose cap skin required to limit its temperature to the maximum specified, in.
TCONTH	-	Thickness of skin required on the top frustum to limit its temperature to the maximum specified, in.
TCORE(I)	-	Honeycomb core height in the Ith bay, in.
TF	-	Sandwich face thickness, in.
TFACE(I)	-	Face thickness in the Ith bay, in.
THETA	-	Frustum half angle, radians
THTA(NF)	-	Half angle of frustum NF, degrees.
TKCIL	-	Skin thickness required to carry the circumferential load, in.
TKHOOP	-	Skin thickness required for hoop stress due to internal pressure, in.

TKMCL	-	Skin thickness required to carry the meridional compressive load, in.
TKSAL	-	Skin thickness required for shear stress due to aerodynamic lift, in.
TMINC	-	Minimum skin thickness to be used in designing a frustum, in.
TMINN	-	Minimum skin thickness to be used in designing the nose cap, in.
TMNC(NF)	-	TMINC for frustum NF, in.
TMP	-	Maximum allowable temperature for both the top frustum and nose cap, °F.
TMPMAX	-	Same as TMP.
TRBMIN	-	Input value of minimum ribbon thickness, in.
TRBN(I)	-	Ribbon thickness for the Ith bay, in.
TRBSTD(JG)	-	Block data values to be used for ribbon thickness if KTRSTD is equal to 1.
TRIBN	-	Ribbon thickness, in.
TRING (I)	-	Thickness of ring material for optimized design of the Ith bay, in.
TSTD	-	Block data of standard sheet metal gauges, in.
T1	-	Thickness of outer face of sandwich, in.
T2	-	Thickness of inner face of sandwich, in.
VCAP	-	Volume of nose cap which is useful for payload. (See HUSE and RUSE), cu. ft.
VCORE	-	Volume enclosed between the two sandwich faces, cu. in.
VGROS	-	Gross volume of nose cap, cu. in.
VGROSS	-	Gross volume of fairing, cu. ft.

VSEG (J)	-	Volume of bay designed for a skin gauge of T (J), cu. in.
VTOT	-	Useful volume of fairing (see DUSE1, DUSE2, HUSE and RUSE), cu. ft.
VUSE	-	Useful volume of frustum (see DUSE1, and DUSE2), cu. ft.
W	-	Average length of a core cell wall, equal to 1/2 (ATB), in.
WBOND (I)	-	Weight of adhesive bond material used to attach faces to core, lbs.
WCAP	-	Weight of nose cap, lbs.
WCONE	-	Frustum weight, lbs.
WFACE (I)	-	Weight of both sandwich faces of the Ith bay, lbs.
WRING (I)	-	Weight of ring for the Ith bay, lbs.
WSEG (I)	-	Weight of the Ith bay, lbs.
WSPLCE (I)	-	Weight of panel splices in the Ith bay, lbs.
WTCORE (I)	-	Weight of core material in the Ith bay, lbs.
WTDEX (I)	-	Weight of bay divided by volume of bay, lbs/cu. in.
WTINFC	-	Weight of inner sandwich face material in the Ith bay, lbs.
WTOT	-	Total fairing weight, lbs.
WTOTFC	-	Weight of outer face material in the Ith bay, lbs.
XBCAP	-	Distance from base of nose cap to center of lift pressure on the nose cap, in.
XL	-	Length of bay, in.
Y	-	Warpage correction factor for thick cores.

APPENDIX C
AERODYNAMIC PRESSURE COEFFICIENTS

When data for the fairing pressure profile is not input, the pressure coefficient at zero angle of attack, CPOO, and the maximum change in pressure coefficient due to angle of attack, CPAA, are computed for each conical frustum in Subroutine AERO. For the purpose of computing these parameters, each frustum is treated as a complete cone with an attached shock. Both CPOO and CPAA will then be uniform in the axial direction for each frustum. In order to construct the pressure profile as described in Section 1.4 and Figure 4, the values computed for CPOO and CPAA for the frustum are assigned to the stations at the ends of the frustum. That is

$$\begin{aligned} \text{CPO (LT)} &= \text{CPOO} \\ \text{CPO (LT+1)} &= \text{CPOO} \\ \text{CPA (LT)} &= \text{CPAA} \\ \text{CPA (LT+1)} &= \text{CPAA} \end{aligned}$$

In which LT is the station at the small diameter of the frustum, and (LT+1) is the station at the large diameter of the frustum.

Using the ground rules indicated above, CPOO can be readily determined through the use of equations developed by Simon and Walter in Reference 2, which agree within a few percent with data presented in Chart 6 of Reference 3 (NACA Report 1135). These equations have been programmed in Subroutine AERO and are used in computing CPOO for each frustum.

When flying at an angle of attack the pressure distribution in the circumferential direction varies with circumferential position. This circumferential pressure distribution is assumed to be sinusoidal (see Figure 5). Since each frustum is treated as a complete cone, the distribution in the axial direction is uniform. The pressure distribution over the entire frustum can now be described by equations specifying the circumferential pressure distribution. These equations, as illustrated in Figure 5, are

$$C_P = C_{PO} - \Delta C_P \sin \phi \tag{C1}$$

$$P = P_{AV} - \Delta P_{MAX} \sin \phi \tag{C2}$$

In which

$$C_{PO} = \text{CPOO}$$

$$\Delta C_P = \text{CPAA}$$

The normal force, ΔF_N , produced on an incremental length, ΔX , by this pressure distribution can be computed as follows:

$$\Delta F_N = \Delta X \int_0^{2\pi} (-P \sin \phi) \left(\frac{D}{2}\right) d\phi \quad (C3)$$

in which D is the diameter of the increment. When the expression for P (Equation C2) is substituted in Equation C3, and the integration performed

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta P_{MAX}) \quad (C4)$$

In which

$$\Delta P_{MAX} = (\Delta C_P) q$$

and

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta C_P) q \quad (C5)$$

The normal force on the same increment of cone can also be computed by using the normal force coefficient, C_N . For a complete cone the normal force, F_N , is expressed by

$$F_N = C_N q A \quad (C6)$$

in which A is the base area of the cone. The incremental normal force, ΔF_N , produced by a short length, ΔX , of this cone is

$$\Delta F_N = C_N q (\Delta A) \quad (C7)$$

in which ΔA is the surface area of the increment projected on the cone base, expressed by

$$\Delta A = \frac{\Delta D}{2} (\pi D) \quad (C8)$$

The change in diameter, ΔD , for a change in length, ΔX , is

$$\Delta D = 2 (\Delta X) \tan \theta \quad (C9)$$

in which θ is the half angle of the cone. Substituting Equations C8 and C9 into C7 yields the following expression for ΔF_N :

$$\Delta F_N = C_N q (\pi D) (\Delta X) \tan \theta \quad (C10)$$

When the two expressions for ΔF_N (Equations C5 and C10) are equated and solved for ΔC_P , the following expression is obtained.

$$\Delta C_P = 2 C_N \tan \theta \quad (C11)$$

In Chart 8 of Reference 3 $C_{N\alpha}$ is plotted as a function of cone half angle θ and Mach number. $C_{N\alpha}$ is defined as

$$C_{N\alpha} = \left(\frac{\partial C_N}{\partial \alpha} \right)_{\alpha = 0} \quad (C12)$$

in which α is the angle of attack. For small angles of attack the following relationship is valid:

$$C_N = (C_{N\alpha})\alpha \quad (C13)$$

For the study for which this computer program was developed maximum loads occur in the neighborhood of Mach 1.5. In this region $C_{N\alpha}$ is not a strong function of Mach number. Therefore, a plot was made of $C_{N\alpha}$ versus θ at Mach 1.5. The points fell on a straight line expressed by the following equation.

$$C_{N\alpha} = 2.03 - 1.2\theta \quad (C14)$$

In Equation C14 both α and θ are expressed in radians. Substituting Equations C13 and C14 into C11 yields the following equation which is used to compute CPAA in Subroutine AERO ($CPAA = \Delta C_P$).

$$\Delta C_P = (2 \tan \theta) (2.03 - 1.2\theta)\alpha \quad (C15)$$

In Equation C15, ΔC_P is the difference between the pressure coefficient on the windward side of the fairing and the pressure coefficient at the neutral position on the fairing.

APPENDIX D
BENDING MOMENTS, AXIAL LOADS AND SHEAR LOADS

In order to design a bay within the nose fairing structure, it is necessary to know the magnitude of the loads to which the bay is subjected. In addition to lateral pressure there are bending moments and axial loads which are used in computing line loads (force per running inch on the circumference) for the bay being analyzed. These computations are performed in Subroutine LOAD using the pressure profile data which was either computed in Subroutine AERO or input to the program.

The pressure profile data consists of a number of points connected by straight line segments as illustrated in Figure 4. In order to compute axial loads, shear loads and bending moments, it is necessary to compute the contribution of each of these pressure profile increments to the total load. In computing these incremental loads the point on the increment nearest the base of the fairing is used as a reference point.

First, an equation is derived to represent the shear force contributed by a pressure profile increment at its reference point. Nomenclature for this derivation is illustrated in Figure D1. The expression for ΔC_P as a function of X/D between locations $(X/D)_1$ and $(X/D)_2$ is as follows:

$$\Delta C_P = \Delta C_{P1} - \frac{\Delta C_{P1} - \Delta C_{P2}}{(X/D)_1 - (X/D)_2} \left[(X/D)_1 - (X/D) \right] \quad (D1)$$

At a specified location ΔC_P is the difference between the pressure coefficient on the windward side of the fairing and the pressure coefficient at the neutral position on the fairing (see Figure 5). The relationship between the variables in Equation D1 and Figure 4 are as follows:

$$\Delta C_{P1} = CPA (LT+1)$$

$$\Delta C_{P2} = CPA (LT)$$

$$(X/D)_1 = XOD (LT+1)$$

$$(X/D)_2 = XOD (LT)$$

Let

$$A_2 = \frac{\Delta C_{P1} - \Delta C_{P2}}{(X/D)_1 - (X/D)_2} \quad (D2)$$

$$A_1 = \Delta C_{P1} - A_2(X/D)_1 \quad (D3)$$

Then, combining Equations D1, D2 and D3

$$\Delta C_P = A_1 + A_2(X/D) \quad (D4)$$

- X/D - Axial location measured from the nose in calibers.
- ΔC_P - Change in pressure coefficient due to angle of attack (identical to CPAA in program listing).
- D - Local diameter of fairing
- θ - Half angle of frustum

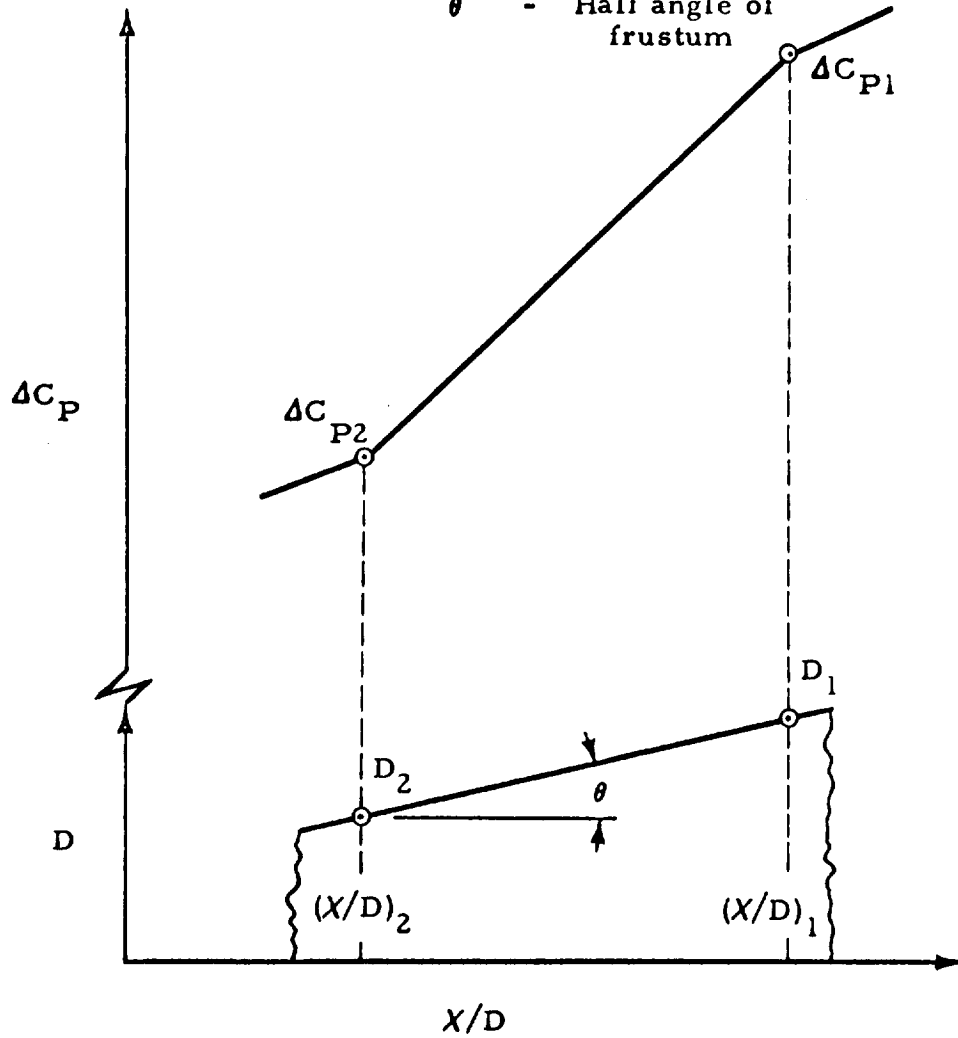


Figure D1 - Nomenclature Used in Derivation of Bending Moment Equation

In a similar manner the following expression can be obtained for the local diameter D .

$$D = A_{D1} + A_{D2}(X/D) \quad (D5)$$

In which

$$A_{D2} = \frac{D_1 - D_2}{(X/D)_1 - (X/D)_2}$$

$$A_{D1} = D_1 - A_{D2}(X/D)_1$$

In Appendix C an equation for the normal force, ΔF_N , for an increment of length, ΔX , was derived for a sinusoidal pressure distribution in the circumferential direction. This equation (Equation C5) is as follows:

$$\Delta F_N = \frac{\pi}{2} D (\Delta X) (\Delta C_P) q$$

in which q is dynamic pressure. Using Equation C5 the running load in the axial direction, w , is expressed as follows:

$$\begin{aligned} w &= \frac{\Delta F_N}{\Delta X} \\ &= \frac{\pi}{2} D (\Delta C_P) q \end{aligned} \quad (D6)$$

By combining Equations D4, D5 and D6, the following expression is obtained for w .

$$w = \frac{\pi}{2} q \left[A_1 A_{D1} + A_1 A_{D2}(X/D) + A_2 A_{D1}(X/D) + A_2 A_{D2}(X/D)^2 \right] \quad (D7)$$

The shear force, v , at any point on the increment is

$$\begin{aligned} v &= \int_{X_1}^X -w \, dX \\ &= -D_{base} \int_{(X/D)_1}^{X/D} w \, d(X/D) \end{aligned} \quad (D8)$$

In which D_{base} is the base diameter of the fairing. When the integration is performed, the following expression is the result.

$$v = \frac{\pi}{2} q D_{\text{base}} \left\{ \beta_1 \left[X/D - (X/D)_2 \right] + \beta_2 \left[(X/D)^2 - (X/D)_2^2 \right] + \beta_3 \left[(X/D)^3 - (X/D)_2^3 \right] \right\} \quad (D9)$$

In which

$$\beta_1 = A_1 A_{D1}$$

$$\beta_2 = \frac{A_1 A_{D2} + A_2 A_{D1}}{2}$$

$$\beta_3 = \frac{A_2 A_{D2}}{3}$$

The incremental shear force, v_i , at the reference point of the i th increment (the point nearest the fairing base), due to aerodynamic pressure acting on the i th increment, is obtained by substituting $(X/D)_1$ for X/D in Equation D9.

The incremental bending moment, M_i , about the reference point of the i th increment is expressed as follows:

$$M_i = \int_{X_2}^{X_1} v \, dx$$

$$= D_{\text{base}} \int_{(X/D)_2}^{(X/D)_1} v \, d(X/D) \quad (D10)$$

By substituting Equation D9 into Equation D10 and performing the integration, the following expression is obtained.

$$\begin{aligned}
 M_i = \frac{\pi}{2} q D_{\text{base}}^2 & \left\{ \frac{\beta_1}{2} \left[(X/D)_1^2 - (X/D)_2^2 \right] \right. \\
 & - \beta_1 (X/D)_2 \left[(X/D)_1 - (X/D)_2 \right] + \frac{\beta_2}{3} \left[(X/D)_1^3 - (X/D)_2^3 \right] \\
 & - \beta_2 (X/D)_2^2 \left[(X/D)_1 - (X/D)_2 \right] + \frac{\beta_3}{4} \left[(X/D)_1^4 - (X/D)_2^4 \right] \\
 & \left. - \beta_3 (X/D)_2^3 \left[(X/D)_1 - (X/D)_2 \right] \right\} \quad (D11)
 \end{aligned}$$

These incremental shear loads and bending moments are now used to compute the shear load, V_{base} , and bending moment, M_{base} , at the base of the fairing.

$$V_{\text{base}} = \sum_{i=1}^I v_i + \text{Contribution of nose cap} \quad (D12)$$

$$M_{\text{base}} = \sum_{i=1}^I (v_i L_i + m_i) + \text{Contribution of nose cap} \quad (D13)$$

in which I is the total number of increments and L_i is the distance from the base of the fairing to the reference point of the i th increment. The shear moment contribution of the nose cap are discussed below.

As design of the fairing moves from the base toward the nose cap, shear and moment contributed by each of the increments of pressure profile are subtracted from the total shear and bending moment. In moving from the reference point of the $(i-1)$ th increment to the reference point of the i th increment the shear and bending moments at the i th reference point are computed as follows:

$$V_i = V_{i-1} - v_{i-1} \quad (D14)$$

$$M_i = M_{i-1} - m_{i-1} - V_i x_{i-1} \quad (D15)$$

in which x is the length of the increment.

Usually the location of the base of a bay will not coincide with the beginning or end of a pressure profile increment. In this case the pressure profile increment is divided at the base of the bay and each part is treated as a complete increment.

Computation of axial loads is handled in much the same manner as computation of shear force. Using nomenclature similar to that used previously for the ΔC_P calculations (see Figure D1) the equation for the pressure coefficient within a pressure profile increment is

$$C_P = B_1 + B_2 D \quad (D16)$$

in which

$$B_1 = C_{P1} - \frac{C_{P1} - C_{P2}}{D_1 - D_2} D_1 \quad (D17)$$

$$B_2 = \frac{C_{P1} - C_{P2}}{D_1 - D_2} \quad (D18)$$

The incremental axial load is

$$\Delta F_{ax} = \int_{D_1}^{D_2} (q C_P - \Delta P) (\pi D) \frac{dD}{2} \quad (D19)$$

in which ΔP is the difference between fairing internal pressure and ambient pressure. When the expression for C_P is substituted into this equation and the integration is performed, the following equation is obtained.

$$\Delta F_{ax} = \frac{\pi}{2} \left[\frac{1}{2} (q B_1 - \Delta P) (D_1^2 - D_2^2) + \frac{1}{3} q B_2 (D_1^3 - D_2^3) \right] \quad (D20)$$

Total axial load at the base of the fairing is computed by summing up the incremental loads plus the drag contributed by the nose cap. As design of the fairing progresses from the base towards the nose cap, increments of axial load are subtracted in a manner similar to that employed in computing shear loads.

When the bending moment and axial load are known at the base of a bay, the circumferential line load can be computed. This is the load per unit length of circumference parallel to the surface of the skin. The axial load places a uniform compressive load on the circumference. The bending moment places a compressive load on the leeward side and a tensile load on the windward side. The line load due to the axial force is

$$(N\phi)_{AX} = \frac{F_{ax}}{\pi D \cos \theta} \quad (D21)$$

in which

- F_{ax} = Axial force at the base of the bay
- D = Diameter at the base of the bay
- θ = Semi-vertex angle of the bay

Using the assumption that the strain in the skin due to bending is proportional to the distance from the neutral plane, the maximum contribution of bending moment to the line load is computed by the following equation:

$$(N\phi)_{BEND} = \frac{M}{\frac{\pi}{4} D^2} \frac{1}{\cos \theta} \quad (D22)$$

When line load due to bending is superimposed on line load due to axial force the total becomes

$$(N\phi)_{WND} = (N\phi)_{AX} - (N\phi)_{BEND} \quad (D23)$$

$$(N\phi)_{LEE} = (N\phi)_{AX} + (N\phi)_{BEND} \quad (D24)$$

The subscript WND indicates windward side, and the subscript LEE indicates the leeward side.

The contribution of the nose cap to axial load, shear force and bending moment are computed by means of the nose cap drag coefficient, C_D , normal force coefficient per radian angle of attack, $C_{N\alpha}$, and \bar{X} , the distance from the base of the nose cap to center of pressure on the normal plane. The reference area for C_D and $C_{N\alpha}$ is the base area of the nose cap. These parameters can be read into the computer or computed in Subroutine LOAD.

The computations for the nose cap C_D in Subroutine LOAD are based on a computed pressure coefficient at the stagnation point, $(C_P)_{stg}$, and a pressure distribution over the nose cap described by the following equation:

$$C_P = (C_P)_{stg} \sin^2 \phi \quad (D25)$$

in which ϕ is the angle between a plane tangent to the nose cap surface and the line of flight.

In order to compute $(C_P)_{stg}$, the pressure at the stagnation point is assumed to be equal to the stagnation pressure downstream from a normal shock with upstream Mach number equal to that of the vehicle. For one-dimensional flow of a perfect gas with constant specific heat and molecular weight the ratio of downstream stagnation pressure, P_o , to upstream static pressure, P_∞ , is expressed by the following equation taken from Reference 11.

$$\frac{P_o}{P_\infty} = \left[\frac{\gamma+1}{2} M^2 \right]^{\frac{\gamma}{\gamma-1}} \left[\frac{2\gamma}{\gamma+1} M^2 - \frac{\gamma-1}{\gamma+1} \right]^{\frac{1}{1-\gamma}} \quad (D26)$$

in which γ is the specific heat ratio of air and M is the Mach number of the vehicle. When $\gamma = 1.4$, Equation D26 reduces to

$$\frac{P_o}{P_\infty} = \frac{166.92 M^7}{(7M^2 - 1)^{2.5}} \quad (D27)$$

An expression for P_∞ derived from basic definitions is as follows:

$$P_\infty = \frac{q}{\frac{1}{2} \gamma M^2} \quad (D28)$$

For air, Equation D28 reduces to

$$P_\infty = \frac{q}{0.7 M^2} \quad (D29)$$

Using the definition for pressure coefficient

$$\begin{aligned} (C_P)_{stg} &= \frac{P_o - P_\infty}{q} \\ &= \left(\frac{P_o}{P_\infty} - 1 \right) \frac{P_\infty}{q} \end{aligned} \quad (D30)$$

Combining Equations D27, D29 and D30 yields the following equation

$$(C_P)_{stg} = \frac{1}{0.7 M^2} \left[\frac{166.92 M^7}{(7M^2 - 1)^{2.5}} - 1 \right] \quad (D31)$$

The pressure coefficient, C_P , at all points on the nose cap is now defined by Equations D25 and D 31. By integrating C_P over the nose cap surface the following expression is obtained for C_D .

$$C_P = \frac{1}{2} (C_P)_{stg} (1 + \sin^2 \theta) \quad (D32)$$

in which θ is the half angle of the top frustum. The axial force contributed by the nose cap is now expressed by the following equation:

$$(\Delta F_{ax})_{CAP} = C_D q A \quad (D33)$$

in which A is the base area of the cap.

For small bluntness ratio (less than 0.2) the bending moment contribution of the nose cap can be approximated by assuming that the nose cap is replaced by a cone having the same half angle as the top frustum. C_{N_α} can then be computed by Equation C14 which is derived in Appendix C.

$$C_{N_\alpha} = 2.03 - 1.2 \theta \quad (D34)$$

Shear force contribution of the nose cap is

$$v_{CAP} = \alpha C_{N_\alpha} q A \quad (D35)$$

For a complete cone the normal force (shear force) acts a point one-third of the distance from the base of the cone to its apex.

$$\bar{X} = \frac{1}{3} \frac{d}{2 \tan \theta} \quad (D36)$$

In which d is the diameter of the nose cap base. The bending moment at the base of the nose cap is expressed as follows:

$$m_{CAP} = \bar{X} v_{CAP} \quad (D37)$$

APPENDIX E
LATERAL PRESSURE

The lateral pressure used to design the fairing is the difference between internal and external surface pressure multiplied by a factor of safety. This pressure is computed on both the windward and leeward sides of the bay, using the pressure profile data and input aerodynamic data.

Pressure coefficients on the windward and leeward sides of the bay are expressed by the following equations.

$$(C_P)_{WND} = C_{PO} + \Delta C_P \quad (E1)$$

$$(C_P)_{LEE} = C_{PO} - \Delta C_P \quad (E2)$$

In which

$(C_P)_{WND}$ = pressure coefficient on windward side

$(C_P)_{LEE}$ = pressure coefficient on leeward side

C_{PO} = pressure coefficient at zero angle of attack

C_P = change in pressure coefficient due to angle of attack

The difference between surface pressure and free-stream pressure is expressed by the following equations.

$$(P_S)_{WND} - P_\infty = (C_P)_{WND} q \quad (E3)$$

$$(P_S)_{LEE} - P_\infty = (C_P)_{LEE} q \quad (E4)$$

In which

P_S = surface pressure

P_∞ = free-stream pressure

q = dynamic pressure

Recall that the difference between internal pressure and free-stream pressure is an input parameter.

$$\Delta P = P_{int} - P_\infty \quad (E5)$$

In which

ΔP = the input value of pressure difference

P_{int} = absolute pressure inside the fairing

Combining Equation E3 with E5 and E4 with E5 the following equations for pressure difference are obtained.

$$(P_S)_{WND} - P_{int} = (C_P)_{WND} q - \Delta P \quad (E6)$$

$$(P_S)_{LEE} - P_{int} = (C_P)_{LEE} q - \Delta P \quad (E7)$$

Design pressures are obtained by multiplying these pressure differences by the factor of safety, FS.

$$(P_{des})_{WND} = FS \left[(C_P)_{WND} q - \Delta P \right] \quad (E8)$$

$$(P_{des})_{LEE} = FS \left[(C_P)_{LEE} q - \Delta P \right] \quad (E9)$$

APPENDIX F
FACE WRINKLING AND DIMPLING

Face wrinkling involves the buckling of one face only. Normally, an analysis is made in a manner similar to a beam on an elastic foundation. The analysis used here (Reference 12) depends upon whether the core is "thin" or "thick". If the core is thin, it is assumed to behave like a group of independent springs with no shearing forces between them. If the core is thick, shear forces between these springs are considered. If $\frac{h}{R} > \frac{2W}{R}$ as determined by the following equation, the core is considered thick.

$$\frac{2W}{R} = \frac{2.496 t_f}{R} \left(\frac{E/G_c}{t_c/a} \right)$$

where:

- R = cylinder mean radius
- t_f = face thickness
- E = elastic modulus of face
- G_c = shear modulus of core
- t_c = core ribbon thickness
- a = free honeycomb-core wall dimension between nodes

For thick cores, wrinkling stress is predicted by

$$\sigma_\omega = 1.66 (E)^{1/3} (G_c)^{2/3} (t_c/a)^{2/3}$$

For thin cores, wrinkling stress is predicted by

$$\sigma_\omega = 1.86 (E)^{1/2} (G_c)^{1/2} (t_c/a)^{1/2} (t_f/h)^{1/2}$$

where:

- h = distance between midplanes of face sheets

The above equations predict wrinkling stress levels if there are no imperfections in the face sheets. To approximately account for the deleterious effect of reasonable imperfections, half of the predicted wrinkling stress level has been used as an allowable stress. Normally, designs having practical ribbon and face thicknesses will not be critical for wrinkling.

Face dimpling considerations limit the maximum size of the core hexagon. From Reference 12, the following equation is used to predict dimpling:

$$\sigma_d = \frac{2t_f^2 E}{a^2(1-\nu^2)}$$

where:

ν = Poisson's ratio of face sheets.

a = Minimum distance between nodes.

One half of the predicted dimpling stress value as determined by the above equation was used as an allowable stress level.

APPENDIX G
STIFFENING RING DESIGN

After the shell portion of a bay has been designed, it is necessary to provide a ring of adequate stiffness to prevent general instability of the composite structure, i.e., to prevent the entire side of the fairing from caving in. This ring is placed at the upper end of the bay. For tapered conical sections, lateral crushing pressure is normally the dominant factor in ring size determination. For these sections, the required moment of inertia of such a ring is expressed by the following equation which was used by Nevins and Helton in a similar study reported in Reference 1.

$$I_{\text{req}} = L_{\text{bay}} \left(\frac{D}{2 \cos \theta} \right)^2 \left(\frac{1}{t} \right)^{1/3} \left[\frac{(P_{\text{des}})_{\text{WND}} D_b}{11.02 E \tan \theta} \right]^{4/3} \quad (G1)$$

in which

- L_{bay} = length of bay, in.
- D = small diameter of bay, in.
- θ = semivertex angle of bay
- t = skin thickness, in.
- $(P_{\text{des}})_{\text{WND}}$ = the crushing pressure on the windward side of the fairing, psi.
- D_b = the base diameter of the fairing, in.
- E = modulus of elasticity of the material, psi.

This equation is a modification of the general stability equation developed by Becker in Reference 5.

For cylindrical sections, axial loads are higher and lateral collapse pressure much lower than in conical sections, hence rings are sized on a different basis. As stated in Section 1.13, the cylindrical section is divided into bays of equal length using stiffening rings of identical cross-section. Skin gauge for the first bay is determined by the same methods used elsewhere in the fairing. Then assuming that the skin gauge and loading determined for the first bay prevail throughout the cylindrical section, the minimum ring cross-section required to prevent general instability is computed by the method of Baruch-Singer adapted for use in this program by A. B. Burns (see Appendix K).

After computing the moment of inertia required of the ring, the ring cross-section which will provide this moment of inertia is selected. The three types of cross-sectional shapes which may be specified are shown in Figure 3. Also to be specified are B/t ratios of the web(s) and flanges. When designing the ring, the computer program selects the smallest standard skin gauge which provides a ring cross-section with moment of inertia equal to or greater than that required, providing that the selected ring has no buckled flanges.

It is necessary to check for flange (or web) buckling because large B/t values may be input, and these large B/t values present a definite possibility of local instability occurring. In making this check, a small (one percent) ovality tolerance was assumed, and bending stress due to this ovality effect are added to the hoop compression stress. The total flange stress thus obtained, is compared to an input flange buckling stress level (FCFB), and if excessive, the ring web thickness is increased as required.

APPENDIX H
STRUCTURAL DESIGN OF SPHERICAL NOSE CAP

The nose cap design is analyzed structurally as an unstiffened, non-shallow spherical cap with uniform shell thickness. The method of analysis used is presented in Section 6.23.1 of Reference 4. From experiments it is observed that non-shallow (the ratio of height to radius is greater than 1/6) spherical caps buckle in the form of a small dimple in some area of the surface of the cap. Therefore, the critical buckling pressure for non-shallow caps is independent of the height to radius ratio, depending only on the radius to thickness ratio and the modulus of elasticity of the shell material.

The equation recommended in Reference 4 is

$$P_{crt} = \frac{0.606 E}{\left(\frac{R}{t}\right)^2 e^{0.04 \sqrt{R/t}}} \quad (H1)$$

In which

P_{crt} = critical buckling pressure

R = nose cap radius

E = modulus of elasticity

t = shell thickness

A trial and error procedure is used in determining the minimum shell thickness required for the nose cap. When R , E and design pressure are known the shell thickness is increased by 0.001 inch increments until P_{crt} is equal to or greater than the design pressure computed for the nose cap.

Maximum design pressure for the nose cap occurs at the stagnation point. Assuming that the pressure on this point is the same as the stagnation pressure downstream from a normal shock, the pressure coefficient, $(C_P)_{stg}$, is expressed by Equation D30, and the design pressure is expressed by the following equation:

$$(P_d)_{CAP} = (FS) \left[q (C_P)_{stg} - \Delta P \right] \quad (H2)$$

in which

FS = factor of safety

q = dynamic pressure

ΔP = internal to free-stream pressure difference

APPENDIX I
EFFECTIVE SHEAR MODULUS OF
HONEYCOMB CELLULAR STRUCTURE

The general instability analysis developed for this program accounts for the effect of transverse shear deformation, hence buckling loads are a function of cone shear stiffness. Core shear stiffness is calculated for aluminum hexagon honeycomb using the analysis of Penzien and Didriksson (Reference 13). The effective shear modulus is

$$G_c = \frac{G \sin\beta (R + \cos\beta)}{\frac{A}{t_r} \left[(1 + R) \sin^2\beta \cos^2\phi + (R + \cos\beta)^2 \sin^2\phi \right]}$$

where:

G = shear modulus of core material

R = B/A (See Figure 1 below)

t_r = thickness of core material

φ = angle between direction of the shear force and ribbon direction

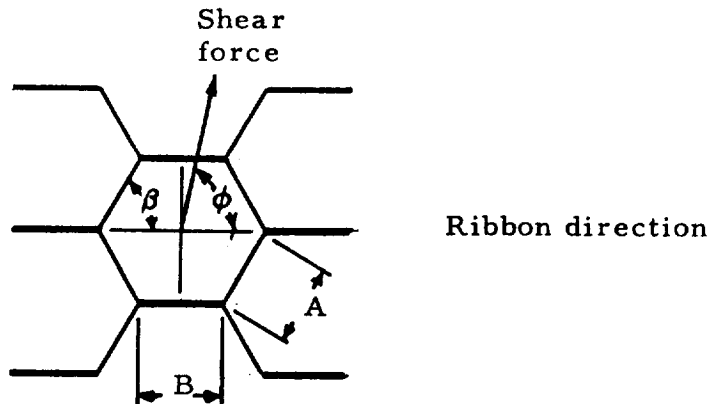


Figure 1 - Geometry of honeycomb cells

A correction factor based on Figure 6 of Reference 13 is also applied to the above equation when the ratio of core height (H) to average cell wall length ($\frac{1}{2}A + \frac{1}{2}B$) is 5 or less.

When the ratio of core height to the quantity $W = 0.5 (A + B)$ is five or less, prevention of warpage increases the effective shear modulus (See Figure 6 of Reference 13). For values of H/W less than five, the effective shear modulus is calculated from the equation

$$G_{\text{ceff}} = G_c (1 + 0.01Y)$$

where:

$$Y = (9.3/(H/W)) - (2.1/(H/W)^2)$$

APPENDIX J
THERMAL ANALYSIS

The maximum temperature reached by the skin due to aerodynamic heating can be controlled by placing a lower limit on the thickness of skin to be used in critical locations. Provisions have been made in the program to place such constraints on both the nose cap and top frustum of the fairing, either by specifying the minimum skin thickness to be used or by specifying the maximum temperature to be reached by the skin.

Design curves for determining the minimum thicknesses have been prepared by LMSC/HREC. These curves and the methods used in generating the data for these curves are presented in Reference 6. The trajectory used in this analysis was a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile circular orbit.

Both laminar and turbulent heating occur during the flight. Laminar flow was assumed to exist when the Reynolds number based on momentum thickness of the local boundary layer was equal to or less than 500. Turbulent flow was assumed to exist at Reynolds numbers above 500. When the flow was laminar, the method of Fay and Riddell (Reference 7) was used together with the laminar heating rate distribution of Lees (Reference 8). When the flow was turbulent, heating rates were calculated by using a method from Reference 9 (Bromberg, Fox and Ackermann). Radiation from the outer surface was also taken into account.

Other assumptions were that the heat flow is one-dimensional, that at any time or location on the fairing the skin temperature is uniform throughout the thickness of the skin, and that the inner surface of the skin is perfectly insulated. These latter assumptions were found to have only a minor effect on the final results.

Maximum temperature constraints are applied to the nose cap and the top frustum. The thickness of material required to limit the maximum temperature of the nose cap is based on heating at the stagnation point of the nose cap, and the thickness required for the top frustum is based on heating on the nose cap at its junction with the top frustum. Thus, the heating data required to establish these constraints can be obtained from a spherical shell.

Several hundred data points were generated for each of the following five materials: aluminum, magnesium, titanium, stainless steel and Lockalloy. Each data point for a specified material is completely described by the following four parameters:

- R = radius of spherical nose cap
- θ = the angle between the line of flight and a plane tangent to the nose cap at the point of interest

T_{\max} = the maximum temperature reached by the skin, °F

t = thickness of the skin

When applying this data to nose fairing design, θ is equal to 90° at the stagnation point and to the half angle of the top frustum at the junction of the nose cap and top frustum. (Note that ϕ in Reference 6 is the complement of θ .)

In order to avoid the necessity of storing all of these data points in the fairing design program, a set of linear algebraic equations which fit the data within a few percent was developed by a technique commonly referred to as multiple regression analysis. A detailed description of multiple regression analysis can be found in many statistical text books such as Reference 10, Chapters 4 and 5.

Two major steps are involved in such an analysis. First, it is necessary to establish the form of the equation relating the variables which describe the data points. This step can be based on intuition and/or a knowledge of the physical laws relating the variables. This equation must be reduced to linear form, which is then referred to as a linear model. The linear model has the general form

$$y = c_1 x_1 + c_2 x_2 + \dots + c_n x_n \quad (J1)$$

The variables y and x_1, x_2, \dots, x_n may be grouped parameters such as $(T_{\max} - 70)$ and T_{\max} / \sqrt{R} . However, these variables must be such that numerical values can be obtained for each variable for each data point.

Having developed a linear model the next step is to determine the set of coefficients (c_1, c_2, \dots, c_n) which give the best fit to the data. This is done by means of the "least squares" curve fit technique. Usually several different linear models are tried in an attempt to curve-fit a given set of data.

For this application two models were developed, one for the stagnation point and the other for the point of tangency between the spherical nose cap and the top frustum. A set of coefficients was computed for each of the two models for each of the five materials, making a total of ten sets of coefficients. The equations and coefficients appear in Subroutine THERML.

The linear model representing heating at the stagnation point is based on laminar and radiative heating theory. It is postulated that the following

relationship is approximately true.

$$\rho C_p t (T_{\max} - 70) = K_1 + \frac{K_2}{\sqrt{R}} (K_3 - T_{\max}) - K_4 \epsilon (T_{\max} + 460)^4 \quad (J2)$$

in which

ρ = density of skin

C_p = specific heat of skin

ϵ = emissivity of skin

The terms in Equation J2 represent the following physical quantities:

$C_p t (T_{\max} - 70)$ = the maximum quantity of heat stored in a unit area of skin during the flight

K_1 = a constant

$\frac{K_2}{\sqrt{R}} (K_3 - T_{\max})$ = convective heat input (laminar flow) to the unit area

$K_4 \epsilon (T_{\max} + 460)^4$ = radiative heat loss from the unit area

When material properties are dropped (coefficients are determined for each material) and when the multiplications are performed Equation J2 reduces to the following linear form.

$$t (T_{\max} - 70) = a_1 + a_2 \frac{1}{\sqrt{R}} + a_3 \frac{T_{\max}}{\sqrt{R}} + a_4 (T_{\max} + 460)^4 \quad (J3)$$

Comparing this to Equation J1

$$y = t (T_{\max} - 70)$$

$$x_1 = 1$$

$$x_2 = \frac{1}{\sqrt{R}}$$

$$x_3 = \frac{T_{\max}}{\sqrt{R}}$$

$$x_4 = (T_{\max} + 460)^4$$

Using the data generated in Reference 6 the coefficients a_1 , a_2 , a_3 and a_4 can be determined by the least-squares technique.

A more complex model is required to represent heating at the point at which the nose cap is tangent to the top frustum. Turbulent flow occurs during part of the flight, and laminar flow occurs during the remainder. It is also necessary to specify the angular location, θ , on the sphere. The following model was postulated:

$$\rho C_p t (T_{\max} - 70) = K_1 + \frac{K_2}{\sqrt{R}} (K_4 + \sin^2 \theta) (K_5 - T_{\max})$$

$$+ \frac{K_3}{R^{0.2}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) - K_6 \epsilon (T_{\max} + 460)^4$$

The additional terms in this equation have the following physical significance:

$$\frac{K_2}{\sqrt{R}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) = \text{convective heat input during laminar flow}$$

$$\frac{K_3}{R^{0.2}} (K_4 + \sin^2 \theta) (K_5 - T_{\max}) = \text{convective heat input during turbulent flow}$$

in which $(K_4 + \sin^2 \theta)$ accounts for pressure variation with angular position on the nose cap. When material properties are dropped and the multiplications are performed the quantities corresponding to the variables in the linear model are as follows:

$$y = t (T_{\max} - 70)$$

$$x_1 = 1$$

$$x_2 = \frac{1}{\sqrt{R}}$$

$$x_3 = \frac{T_{\max}}{\sqrt{R}}$$

$$x_4 = (T_{\max} + 460)^4$$

$$x_5 = \frac{\sin^2 \theta}{\sqrt{R}}$$

$$x_6 = \frac{T_{\max} \sin^2 \theta}{\sqrt{R}}$$

$$x_7 = \frac{1}{R^{0.2}}$$

$$x_8 = \frac{T_{\max}}{R^{0.2}}$$

$$x_9 = \frac{\sin^2 \theta}{R^{0.2}}$$

$$x_{10} = \frac{T_{\max} \sin^2 \theta}{R^{0.2}}$$

Note that y and x_1 , x_2 , x_3 , and x_4 are identical for both the stagnation point and tangency point models.

Coefficients were determined for each of the five materials at both the stagnation point and the tangency point. A summary of pertinent information about the curve fit is presented in Table J1.

The coefficients stored in Subroutine THERML were determined for a nominal two-stage Saturn V ascent trajectory to a 100 nautical mile orbit. If the subroutine is to be used for trajectories which differ greatly from this trajectory it would be advisable to determine a new set of coefficients based on thermal data for the new trajectory. The linear models used in Subroutine THERML will probably be valid for a wide range of trajectories.

TABLE JI
THERMAL CURVE-FIT INFORMATION

Location	Material	Range of Data				No. of Data Points	Range of Errors in Skin Thickness, Percent	
		Temperature, °F		Skin Thickness, in			Negative	Positive
		Low	High	Low	High			
Stagnation Point	Aluminum	366	987	0.050	0.400	-0.66	0.50	
	Magnesium	388	1000	0.060	0.500	-0.93	0.42	
	Titanium	579	1234	0.040	0.250	-0.39	0.30	
	Stainless Steel	746	1361	0.025	0.080	-0.91	1.26	
	Lockalloy	369	1175	0.035	0.300	-1.50	0.78	
Tangency Point	Aluminum	404	1091	0.025	0.200	-3.44	3.84	
	Magnesium	396	1132	0.020	0.300	-7.00	4.81	
	Titanium	475	1365	0.015	0.200	-3.49	3.79	
	Stainless Steel	647	1147	0.0125	0.060	-4.55	7.84	
	Lockalloy	395	1234	0.025	0.150	-6.03	7.20	

NOTES:

Nose cap radius ranges from 13 to 52 inches for both locations and all materials.

Angular location of the tangency point (half angle of the top frustum) ranges from 0 to 45 degrees for all materials for the tangency point data.

APPENDIX K
GENERAL INSTABILITY ANALYSIS OF
HONEYCOMB SANDWICH CYLINDERS

by

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GENERAL INSTABILITY ANALYSIS OF HONEYCOMB SANDWICH CYLINDERS

Summary

In the establishment of the general instability load for a sandwich shell, the practical method of analysis which was recommended in Ref. 1 is used. This method uses the classical buckling load as an upper bound and the minimum postbuckling load as a lower bound to the critical load of the shell. An empirical reduction factor is applied to the upper bound in such a manner that the critical load will not fall below the lower bound. For the case of pure axial compression upper as well as lower bound analyses are available in Ref. 2. The classical buckling load analysis is modified here to include the effects of lateral pressure but the reduction factor has been chosen in a more expedient manner.

Classical Buckling Load

For sandwich cylinders under pure axial compression it was found in Ref. 3 that buckling can occur either in an axisymmetrical or in a nonsymmetrical mode, the former being critical for cylinders with weaker cores. It may be shown that the critical axial load corresponding to symmetrical buckling is independent of the lateral pressure. On the other hand the nonsymmetrical buckling load is reduced in the presence of an external pressure. According to Ref. 2, for the symmetrical pattern:

$$\bar{N} = (1 + F)z + 1/(4z) - 4z^2/[4z + (1/G)]$$

Here \bar{N} is to be minimized with respect to the wavelength parameter z .

After addition of the influence of external pressure, we have for the non-symmetrical buckling mode:

$$\bar{N} = (1 + F)(1 + \beta^2)^2/(4\beta^2/\eta) + (\beta^2/\eta)/(1 + \beta^2)^2 + (b_2^2 + \mu c_2^2)/(4\beta^2/\bar{G}) + s_2/(8\beta^2/\eta) - \bar{p}/\beta^2$$

where

$$s_2 = 2[\beta^2 b_1^2 + \mu^2 c_2^2 + 2\nu\mu\beta b_2 c_2 + 2(1 - \nu)(b_2 + \mu\beta c_2)^2 + 4[b_2\beta(2 - \nu + \beta^2) + c_2\mu\{1 + (2 - \nu)\beta^2\}]]$$

$$b_1 = -8\beta^3/[1/(\bar{G}\eta) + 4\beta^2] \tag{1}$$

$$b_2 = -\beta(1 + \beta^2)\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2\}/[\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2 + \mu\}\{1/(\bar{G}\eta) + 1 - \nu + \beta^2\} - \mu\beta^2]$$

$$c_2 = -(1 + \beta^2)\{1/(\bar{G}\eta) + 1 - \nu\}/[\{1/(\bar{G}\eta) + (1 - \nu)\mu\beta^2 + \mu\}\{1/(\bar{G}\eta) + 1 - \nu + \beta^2\} - \mu\beta^2]$$

The nonsymmetrical buckling load is found through minimization of \bar{N} with respect to the wavelength parameters η and β . These parameters can only take on the specific values which correspond to an integer value for the number of waves in the circumferential and the number of halfwaves in the axial direction. However, we will only apply here the restriction that the axial halfwave length cannot be larger than the shell length. That is

$$\eta \leq \alpha/\beta^2$$

where, for equal face sheets

$$\alpha = \pi^2 \sqrt{1 - \nu^2} Rh/L^2$$

In the computer program the symmetrical buckling load is first determined. The equation $\partial \bar{N} / \partial z = 0$ is solved by use of the Newton-Raphson method. As initial estimates for z are used

$$z = \begin{cases} 1/(2 - 4\bar{G}) & \text{if } \bar{G} < .5 \\ 1/\sqrt{4F} & \text{if } \bar{G} \geq .5 \end{cases}$$

It appears that the Newton-Raphson method would be most efficient whenever reasonably close estimates are available. In the procedure of optimization close estimates are generally available in the form of previous solutions. In such cases the Newton-Raphson method is used directly. However, when a new case is started close initial estimates are not available and the procedure is modified as follows. The initial estimates are

$$\beta = 1$$

$$\eta = \begin{cases} 2\alpha & \text{if } \alpha \geq 1 \\ 1 & \text{if } \alpha < 1 \end{cases}$$

These estimates are improved by use of the steepest descent method before they are used in the Newton-Raphson method. If during iteration η becomes less than α/β^2 this indicates that buckling will be with one halfwave in the axial direction. Consequently we set $\eta = \alpha/\beta^2$ and minimize with respect to β only. After minimization the proper value of \bar{N} is computed and the classical buckling load is chosen as the lowest of the symmetrical and non-symmetrical buckling loads.

Reduction Factor

Inclusion of lateral pressure in the lower bound analysis would be a major undertaking and was not considered necessary for the purpose of this study. For monocoque cylinders it was shown in Ref. 3 that the ratio between

lower and upper bound increases when an internal pressure is added. It seems reasonable to assume that this is the case also for sandwich cylinders. It is well known also that for cylinders under external pressure the agreement between theory and test is much better than it is for cylinders under axial compression. Consequently it seems safe to assume that a conservative estimate of the reduction factor will be obtained if the effect of lateral pressure is neglected. It is expected that the shells in this investigation will be primarily subjected to axial compression and thus the conservatism involved will be moderate.

When the effects of lateral pressure are neglected, the ratio N_{min}/N_{cl} may be obtained from Ref. 2. A number of values of N_{min}/N_{cl} are read from the curves of that analysis and stored in the computer. These values correspond to different shell parameter combinations in such a manner that intermediate results can be obtained easily through interpolation.

According to Ref. 1:

$$N_x = N_{cl} \left[\frac{N_{min}}{N_{cl}} + c \left(1 - \frac{N_{min}}{N_{cl}} \right) \right]$$

where

$$c = \frac{\varphi - 0.12}{0.88} \tag{2}$$

$$\varphi = \begin{cases} 1.0 & \text{for } (R/t)_e \leq 33 \\ 6.48 / [(R/t)_e] & \text{for } (R/t)_e > 33 \end{cases}$$

For equal face sheets

$$(R/t)_e = R / \sqrt{t^2 + 3h^2}$$

Nomenclature

R	mean radius of cylinder
E	Young's modulus of face sheets
ν	Poisson's ratio of face sheets
t	thickness of face sheets
h	distance between midplanes of face sheets
G_1	shear modulus of core in axial direction
G_2	shear modulus of core in circumferential direction
ℓ_x, ℓ_y	axial and circumferential half-wave lengths
p	external pressure
N	axial compressive load per unit width
\bar{N}	$RN(1 - \nu^2)^{1/2}/(2Eht)$
\bar{p}	$R^2 p(1 - \nu^2)^{1/2}/(Eht)$
F	$(t/h)^2/3$
\bar{G}	$1/2(E/G_1)(t/R)(1 - t/h)(1 - \nu^2)^{-1/2}$
μ	G_1/G_2
α	$\pi^2(1 - \nu^2)^{1/2}(Rh/L^2)$
z	β^2/η
β	ℓ_y/ℓ_x
η	$\pi^2(1 - \nu^2)^{1/2}(Rh/\ell_y^2)$
N_{\min}/N_{cl}	ratio between lower and upper bounds for critical load
c	see Eq. 2
φ	see Eq. 2
$b_1, b_2,$ c_2, s_2	see Eq. 1
$(R/t)_e$	$R/\sqrt{t^2 + 3h^2}$

Subroutine Symbols

CRG1 critical buckling line load (pounds/in)

E Young's modulus of facing sheets

G1 Effective shear modulus of core in axial direction

G2 Effective shear modulus of core in circumferential direction

HT distance between midplanes of sandwich facing sheets

K optional control; used to speed up calculations for classical
buckling load if desired

P collapse pressure (psi); burst pressure is entered as
negative quantity

RD cylinder mean radius

T1, T2 thicknesses of face sheets

XL cylinder length

XNU Poisson's ratio of facings material

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