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WYLE LABORATORIES - RESEARCH STAFF

TECHNICAL MEMORANDUM 68-13

TEST PLAN FOR

MARL-EXPERIMENTAL SHROUD TEST PROGRAM

By

G.C. Kao

Work Performed Under Contract NAS8-21260



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RESEARCH DIVISION, HUNTSVILLE FACILITY

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TABLE OF CONTENTS

		Page						
LIST	T OF TABLES	iii						
LIST	T OF FIGURES	iv						
Ι.	PURPOSE	1						
II.	CRITICALITY OF RESULTS AND MANPOWER ESTIM	ATE 1						
III.	TEST SPECIMEN	1						
ï۷.	TEST SETUP	۱						
۷.	TEST PROCEDURE							
VI.	DATA REQUIREMENTS							
VII.	DISPOSITION OF SPECIMEN							
VIII.	REPORT REQUIREMENTS	4						
APF	PENDIX A DERIVATION OF OUTPUT SIGNALS O GAGE TRANSDUCERS	F STRAIN- 22						

1'e

LIST OF TABLES

able		Page
Ι	STRAIN-GAGE TRANSDUCER REQUIREMENTS	5
II	ACCELEROMETER REQUIREMENTS	6
Ш	MICROPHONE REQUIREMENTS	8
ΙV	CROSS-CORRELATION DATA REQUIREMENTS FOR STRAIN GAGE SIGNALS	9
V	CROSS-CORRELATION DATA REQUIREMENTS FOR ACCELEROMETER AND STRAIN GAGE TRANSDUCER SIGNALS	10
VI	CROSS-CORRELATION DATA REDUCTION REQUIREMENTS FOR MICROPHONE, STRAIN GAGE TRANSDUCER AND ACCELEROMETER SIGNALS	11

LIST OF FIGURES

Figure		Page
1	Schematic of the Experimental Shroud	12
2	Sectional View of the Experimental Shroud–Payload Structure and Locations of Strain Gage Transducers	13
3	Proposed SPL Spectra Envelope for Experimental Shroud Testing Program	14
4	Block Diagram of the Transducer System	15
5	Test Setup	16
6	Locations of Strain Gage Transducers for Measuring Axial Load and Schematic of the Transducer Circuit (51 through S8 inclusive)	17
7	Arrangement of Strain Gage Transducers for Measuring Lateral Load and Schematic of the Transducer Circuit	18
·8	Locations of Accelerometers	19
9	Desired Locations for Triaxial and Radial Accelerometers	20
10	Locations of Microphones	21

I. PURPOSE

The purpose of this test will be to acquire acoustic and vibration data for the experimental shroud and the attached payload, so that more accurate specifications can be made to define noise attenuation, transmission loss and vibratory loads on the shroudpayload types of structures.

II. CRITICALITY OF RESULTS AND MANPOWER ESTIMATE

The results of this test are needed to substantiate the theoretical estimates of large payload response to shroud vibration induced acoustically. It is estimated that this program will require approximately 1850 manhours.

III. TEST SPECIMEN

The test specimen consists of a cylindrical shroud and a simulated planetary vehicle and its mounting structure. Figure 1 shows the structural configuration of the test specimen.

The shroud is made of six segments of aluminum foil honeycomb with aluminum face sheets. The average surface density of the aluminum structure is 0.0088 pounds per square inch. The assembled shroud is 25 feet high, 21.66 feet in diameter, and weighs approximately 3337 pounds.

The simulated payload consists of a cylindrical tank with an overall length of 156 inches. This tank will be filled with water to simulate the weight of a planetary vehicle during tests. The weight of the simulated payload is approximately 19450 lbs, and is supported by an eight-point mounting structure as shown in Figure 2.

The experimental shroud is equipped with a top cover which has a hoisting bracket at its center for lifting purposes, and a bottom cover which provides a supporting surface on top of the MARL. These covers are equipped with attachment points for cables which secure the shroud to the ground.

The total weight of the test specimen, including shroud, simulated planetary vehicle, top cover, bottom cover, and mounting fixture, is approximately 30,000 pounds.

IV. TEST SETUP

After the test specimen has been completely assembled, it will be lifted on top of the test fixture (MARL) and fastened properly so that it can be transported to the desired locations for acquiring vibro-acoustic data. During each test, the experimental shroud should be oriented with its 0° reference axis toward the center of firing and be placed in a location where the spectrum of sound pressure levels approximates the spectrum

envelope as specified in Figure 3. The specimen must be securely fastened to the ground by cables, as shown in Figure 4, to maintain lateral stability. A check on the leveling of the specimen shall be made prior to each firing. Tires on MARL shall be inflated to a minimum pressure to provide ground vibration isolation.

A minimum of four (4) exposures will be required for this test program.

V. TEST PROCEDURE

Three types of transducers will be used in the data acquisition system, namely:

- a) Strain Gage Transducer
- b) Accelerometer, and
- c) Microphone.

The above transducers shall be connected to the data recording system according to the block diagram as shown in Figure 5. A total of 60 channels is requested. The breakdown on the data channels are as follows:

Microphones	5700 - Auj	18 channels
Accelerometers		30 channels
Strain Gage Transducers	-	12 channels

The location of each transducer is identified by the station number in the longitudinal direction and by the angle θ in the circumferential direction and by the radius, R, in the radial direction, (as shown in Figures 1 and 2, respectively).

Strain-gage transducers are used to measure forces and lateral bending stresses acting on support linkages. The axial forces are measured by transducers S1 through S8; the lateral bending stresses are measured by transducers S9 through S12. The requirements for strain gage transducers are specified in Table I, and the locations are identified by Figures 2, 6 and 7. Derivation of output signals of the strain gage transducers is presented in Appendix A.

The accelerometer requirements are specified in Table II. There are five triaxial accelerometers to be mounted on the test structure and are identified by the following accelerometer group:

(A28L,	A29R,	A30T)
(A31L,	A32R,	A33T)
(A34L,	A35R,	A36T)
(A37L,	A38R,	A39T)
(A40L,	A41R,	A42T)

where L, R, T, represent, respectively, the directions of the longitudinal, radial and tangential accelerations.

Accelerometers A24, A25, A26L and A27L are to be mounted on the simulated payload. The remainder of the accelerometers are to be mounted in the radial direction on the inner wall of the shroud. Detailed locations of these accelerometers are shown in Figure 8.

The microphone requirements are specified in Table III, These microphones can be classified as:

- a) Wall Microphone, and
- b) Internal Microphone.

The wall microphones are to be mounted on the shroud with diaphragms set flush to the exterior wall of the shroud. The internal microphones are to be mounted in the approximate locations shown in Figure 10. Final positions of the microphones shall be measured to within ± 0.5 in.

After the instrumentation has been installed, the test specimen must be visually inspected. Any holes in the covers, around the edge of the covers, or in the shroud must be filled with an appropriate compound. Before the test, all instrumentation must be calibrated with standards traceable to the National Bureau of Standards.

During the actual test duration, data must be continuously recorded. After the test, the instrumentation and test specimen must be visually inspected. Any damage must be noted for future reference.

VI. DATA REQUIREMENTS

The oscillograph records and reduced taped data will constitute the date from this test. The oscillograph records will be reviewed to select slice times for digital computer analysis. The following data outputs are requested:

- 1. Standard RAVAN output for all measurements; and
- 2. The cross-correlation of data channels of strain gage transducers, accelerometers and microphones, as specified in Tables IV, V and VI, respectively. This should include:
 - a) Cross-correlation functions,
 - b) Cross-spectral density,
 - c) Phase differences, and
 - d) Coherence function.

Data reduction for the cross-correlation functions shall be performed in the frequency range of $10 \sim 2000$ Hz.

In order to provide valid cross-correlation information for the acquired data, all correlated data channels for the recording system shall be phase-matched.

The maximum phase difference between any two correlated channels shall not exceed ± 5 degrees.

VII. DISPOSITION OF SPECIMEN

The entire test assembly, including top and bottom covers, experimental shroud, simulated payload and supporting structure and MARL, shall be returned to MSFC c/o R-P and VE-SVR.

VIII, REPORT REQUIREMENTS

The test report should include the following: (1) detailed description of the instrumentation used, as well as a description of the calibration procedures; (2) locations of the instrumentation shown by photographs; (3) photographs of the overall test specimen; (4) a detailed description of any damage incurred during the test; and (5) any other items pertinent to the test. The report should be completed within three weeks after the test has been run. TABLE I

STRAIN-GAGE TRANSDUCER REQUIREMENTS

Law States

	Remarks	Upper Linkage	Lower Linkage	Upýer Linkage	Łower Linkage	Upper Linkage	Lower Linkage	Upper Linkage	Lower Linkage	Center, Upper Link.	Center, Lower Link.	Center, Upper Link.	Center, Lower Link.
	Figure No.	2,6	=	1	1	7	=	#	=	2,7	=	2	2
	R(In.)	N.A.	1	=	2	=	1	5	=	3	5	=	=
Location	οθ	0	0	0	0	90	8	180	180	0	0	80	180
	Sta.	N.A.	11	=		11	300 200	=	=	=	=		=
Freq.	Range (Hz)	10~2000	1	=	7	11	11	11	-	=	=		1
	Amplit. µ.in./in.	14000	I	11	=	1	R				=	=	11
	Meas. Type	Strain	II.	18	I	H	z	1	7	1	=	=	
	Meas. No.	S1	52	S3	S4	S5	S6	S7	S8	59	S 10	511	SI2

N.A. – Not applicable

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TABLE II

ACCELEROMETER REQUIREMENTS

Locati Sta. θ ^o
0 280
280
215
215
150
150
150
85
85
20
20
135
135
135
135
150

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TABLE II (Concluded)

ACCELEROMETER REQUIREMENTS

													The second second		_
	Remarks									Mounted on Payload	1	Η	H	=	
	Figure No.	8,9	11	п	R	=	=	=	1	=	5	11	11	II	1
	R(In.)	N.A.	=	=	E	=	11 12		=	=	=	=	8	1	=
Location	θο	0	0	60	06	06	270	270	270	ł	ļ	ł	G	1	ł
	Star.	150	150	150	150	150	150	150	150	216.5	216.5	216.5	60.5	60.5	60.5
Freq.	Range (Hz)	10~2000	=	=	=	11	=	=	=	=	=	=	=	=	=
	Amplit. (g)	± 40	=	=	=	=	=	=	=	± 30	=	=	=	=	=
	Meas. Type	Accel.	=	=	=	=	=	=	=	=	=	ž	=	=	=
	Meas. No.	A29R	A30T	A31L	A32R	A33T	A34L	A35R	A36T	A37L	A38R	A39T	A40L	A41R	A42T
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TABLE III MICROPHONE REQUIREMENTS

		T		Т	Т		Τ	1	T	Ī									
	Remarks	Internal Microphone	Π	н	1	=	=-	п	=	Ξ	1	=	Wall Microphone	н	H	Ξ	=	=	н
	Figure No.	10	=	=	=	=	=	н	u	11	=	=	=	2	8	11	=	1	
	R(In.)	0	30	128.5	=	Ξ	80	=	=	0	50	65	130	=	=	=	5	=	=
Location	θο	I	06م	0	90	180	270	45	135	ł	225	315	315	225	0	60	270	180	0
	Sta.	290 .	225	140	140	1,40	75	75	75	10	185	115	225	225	140	140	140	10	185
Freq.	Range (Hz)	10~ 10, 000	=	=	=	=	8	=	=	=	=	=	=	11	=	=	5	=	=
and a second	Amplit. dB	145	=	=	16	-	п	=	1	=	11	=	155	=	=	=	5	=	11
	Meas. Type	SPL		=	=		=	1	=	11	=		=	=	=	=	=	=	=
	Meas. No.	M43	M44	M45	M46	M47	M48	M49	M50	M51	M52	M53	M54	M55	M56	M57	M58	M59	M60

8

2

TABLE IV

Data Channel	To Be Correlated with
S1	S2
	\$3
	S4
	59
S4	\$10
S5	\$6
	S11
S8	\$7
	S12

CROSS-CORRELATION DATA REQUIREMENTS FOR STRAIN GAGE SIGNALS

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TABLE V

CROSS-CORRELATION DATA REQUIREMENTS FOR ACCELEROMETER AND STRAIN GAGE TRANSDUCER SIGNALS

Data Channel	To Be Correlated with
A17	A29R
	A18
	A19
	A13
	A22
	A24
	A26
	S1
	S2
	S9
	S 10
A29R	A24
	A26
	A38R
	A41R
A28L	A37L
	A40L

TABLE VI

CROSS-CORRELATION DATA REDUCTION REQUIREMENTS FOR MICROPHONE, STRAIN GAGE TRANSDUCER AND ACCELEROMETER SIGNALS

Data Channel	To Be Correlated with
M56	M57
	M58
	M60
	M45
	A17
	A 29R
	S1
	S9
	S 10







Figure 2. Sectional View of the Experimental Shroud–Payload Structure and Locations of Strain Gage Transducers











Figure 5 Test Setup







Center of Linkage

a. Strain Gage Location For S9, and S11

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Section D-D

b. Strain Gage Location for S10, and S12



- c. Schematic of Transducer Circuit
- Figure 7. Arrangement of Strain Gage Transducers for Measuring Lateral Load and Schematic of the Transducer Circuit





Figure 8. Locations of Accelerometers 19





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Figure 10. Locations of Microphones

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APPENDIX A

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DERIVATION OF OUTPUT SIGNALS OF STRAIN-GAGE TRANSDUCERS

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APPENDIX A

DERIVATION OF OUTPUT SIGNALS OF STRAIN-GAGE TRANSDUCERS

The output signal, E_0 , of the strain-gage transducers as shown in Figures 6-a and 7-b can be derived as follows:

Let $I_o = Output current$ $R_g = Output resistance or gage resistance.$

Then it can be shown that *

$$I_{o} = \frac{E(R_{2}R_{4} - R_{1}R_{3})}{R_{2}(R_{1} + R_{4})(R_{g} + R_{3} + R_{4}) + R_{1}R_{3}R_{4} - R_{2}R_{4}^{2} + R_{g}R_{3}(R_{1} + R_{4})}$$
(1)

and the output signal E_o is

$$E_{o} = I_{g} R_{g}$$
(2)

Assume that all gages have the same nominal resistance R and by allowing resistance changes in all four legs of the Weatston bridge, Equation (2) becomes:

$$E_{o} = \frac{ER_{g}}{4(R+R_{g})} \left(\frac{\Delta R_{1}}{R_{1}} - \frac{\Delta R_{2}}{R_{2}} + \frac{\Delta R_{3}}{R_{3}} - \frac{\Delta R_{4}}{R_{4}} \right)$$
(3)

For the strain-gage transducer as shown in Figure 6-b, the changes in dummy gages R_2 and R_4 will be negligible, therefore we may set

$$\Delta R_2 = \Delta R_4 = 0,$$

* Perry, C.C. and Lissner, H.R., "The Strain Gage Primer," McGraw-Hill Book Co. 1955.

hence, the output signal is

$$E_{o} \simeq \frac{ER_{g}}{4(R + R_{g})} \left(\frac{\Delta R_{1}}{R_{1}} + \frac{\Delta R_{3}}{R_{3}} \right)$$
(4)

It is evident that signals introduced by bending stresses will be automatically eliminated from the strain-gage arrangement, as shown in Figure 6-b, therefore, only the axial force will be sensed.

The transducer, as shown in Figure 7-b, is used to measure vibratory bending stresses. Gages R_3 and R_4 are dummy gages. By neglecting changes in R_3 and R_4 , the output signal in this case is:

$$E_{o} \simeq \frac{ER_{g}}{4(R+R_{g})} \left(\frac{\Delta R_{1}}{R_{1}} - \frac{\Delta R_{2}}{R_{2}} \right) \qquad (5)$$

Since ΔR_1 and ΔR_2 due to bending will always have opposite signs, Equation 5 will yield directly the vibratory bending stresses.