## General Disclaimer

## One or more of the Following Statements may affect this Document

- This document has been reproduced from the best copy furnished by the organizational source. It is being released in the interest of making available as much information as possible.
- This document may contain data, which exceeds the sheet parameters. It was furnished in this condition by the organizational source and is the best copy available.
- This document may contain tone-on-tone or color graphs, charts and/or pictures, which have been reproduced in black and white.
- This document is paginated as submitted by the original source.
- Portions of this document are not fully legible due to the historical nature of some of the material. However, it is the best reproduction available from the original submission.

SOME ASPECTS OF
FATIGUE CRACK PROPAGATION
by
Richard Roberts and John J. Kibler


Prepared under Grant No. NGR-39-007-011 and under NSF Grant GK-1225 by LEHIGH UNIVERSITY Bethlehem, Pa. for
NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

FATIGUE CRACK PROPAGATION

by Richard Roberts and John J. Kibler Lehigh University, Bethlehem, Pa.

## SUMMARY

The models of fatigue crack propagation proposed by Forman et al and Roberts and Erdogan were studied in this paper. By applying these models to existing data in the literature for thin 2024-T3 and 7075-T6 aluminum Elates subjected to fluctuating tensile loads, it was found that both models gave comparable results when one considered just a gross correlation of the experimental data. By modifying Forman's model to incorporate the ideas of Roberts and Erdogan, a model was produced which appeared to be a more rational basis for studying the problem of fatigue crack propagation in thin plates and shells subjected to tensile loads, bending loads, or a combination of both. This fact was demonstrated for the case of thin plates subjected to fluctuating bending loads and for the case of thin cylindrical shells subjected to fluctuating internal pressure.

This paper also presents the large quantity of data celating the rate of fatigue crack propagation in thin plates subjected to fluctuating bending loads collected at Lehigh University.

Recent work (refs. 1, 2, and 3) has clearly shown that the rate of fatigue crack propagation in thin sheets subjected to either fluctuating tensile loads or fluctuating bending loads is related to the stress-intensity factor.

It is the purpose of this paper to evaluate the fatigue crack propagation models of Forman et al (ref. 2) and Roberts and Erdogan (ref. 3). This evaluation will be limited to two aluminum alloys, 2024-T3 and 7075-T6. The primary source of data for fatigue crack propagation due to plane extension used in this evaluation will be the work of Donaldson and Anderson (ref. 4) and the work of Schijve and his co-workers (ref. 5). The data relating the effect of bending loads on fatigue crack propagation comes from the extensive work performed at Lehigh University during the past few years.

Since the large quantity of data gathered at Lehigh University relating the rate of fatigue crack propagation to fluctuating bending loads is not readily available in the literature, it is the secondary purpose of this paper to present these data in a convenient form whereby other researchers may make use of these data. To date, none of
the raw data and only a partial analysis of some of the data appears in the literature (refs. 3 and 6).

## FATIGUE CRACK PROPAGATION MODELS

Many functional forms hare been proposed to represent the relationship between the rate of fatigue crack propagation and various physical parameters.

Paris and Erdogan (ref. 1) proposed the following relationship between the rate of fatigue crack propagation and the stress-intensity factor amplitude, $\Delta K$ :

$$
\begin{equation*}
\frac{d a}{d N}=A(\Delta K)^{m} \tag{1}
\end{equation*}
$$

where $a$ is the half crack length for a center-cracked specimen, $N$ is the number of load cycles, $\Delta K$ is the stressintensity factor amplitude defined herein as $\Delta K=\left(K_{\max }-\right.$ $K_{\min } / 2$, A is a constant which must be determined for each material, mean load, etc., and $m$ is a numerical exponent. Paris and Erdogan found the general trend of the data indicated that $m \cong 4$.

Forman et al (ref. 2) argued that a correct crack growth law should include the criterion that the crack growth rate approach infinity as the maximum value of the

level for rapid fracture, $K_{c}$. Thus by modifying the model of Faris and Erdogan, equation 1 , Forman et al proposed an equation of the form

$$
\begin{equation*}
\frac{d a}{d N}=\frac{B(\Delta K)^{n}}{(1-R) K_{c}-2 \Delta K} \tag{2}
\end{equation*}
$$

where $R=K_{\text {min }} / K_{\text {max }}, K_{c}$ is the critical stress-intensity factor for rapid fracture, $B$ is a material constant, and n is a numerical exponent.

In evaluating their model, Forman et al used the data found in (refs. 4, 5, and 7). For the aluminum alloys 2024-T3 and 7075-T6, they found $n \equiv 3$. They also found the values of $K_{c}$ which best fit the data were in reasonable agreement with published values of $K_{c}$.

Erdogan and Roberts (refs. 3 and 6) proposed that the rate of fatigue crack propagation is probably more fundamentally related to the size of the plastic zone ahead of and in the plane of the fatigue crack. This led to an equation of the form

$$
\begin{equation*}
\frac{d a}{d N}=c\left(K_{\max }\right)^{p}(\Delta K) q \tag{3}
\end{equation*}
$$

where $C$ is a material constant, and $p$ and $q$ are numerical exponents. They found for tensile and bending loads that the values of $p$ and $q$ in equation (3), for 2024-T3 and

7075-T6 aluminum alloss, could be approximated as $p=q \cong 2$.

## EXPERIMENTAL RESULTS

The results of two experimental programs are reported in this paper. In the first program, 2024-T3 and 7075-T6 aluminum plates which sontained a central through crack were subjected to fluctueting cylindrical bending loads. Reference (6) gives a detailed description of the testing equipment and procedures used for collecting the data. Appendix $A$ contains the values of the crack length, number of cycles, specimen thickness, load levels, etc., for these tests.

The second program consisted of subjecting 2024-T3 bare aluminum plates containing a central through craci to a combined static tensile load and a cyclic transverse bending load. Reference (12) gives a detailed description of the testing equipment and procedures. The res.alts obtained from the combined loading tests are given in Appendix B.

The mechanical properties of the combined loading specimens and the cylindrical bending specimens are given in Appendix C. The values listed in this appendix are the
average values for four test specimens. These mechanical properties were determined in accordance with the appropriate ASTM standard for sheet material.

## DISCUSSION

The bending data given in the Appendices will not be discussed in detail in this paper. These data will be used solely as a means of comparing the models of Forman et al and Roberts and Erdogan.

The quantity $\beta$ was introduced into equations
and (3) before making the comparison of these equations. $B$ is defined as $\beta=K_{\text {mean }} / \Delta K$ where $K_{\text {mean }}=\left(K_{\text {max }}+K_{\text {min }}\right) / 2$. Thus equations (2) and (3) can be written as

$$
\begin{equation*}
\frac{d a}{d N}=\frac{B(1+B)(\Delta K)^{n}}{K_{c}-(1+B) \Delta K} \tag{4}
\end{equation*}
$$

and

$$
\begin{equation*}
\frac{d a}{d N}=c(1+B)^{p}(\Delta K) p+q \tag{5}
\end{equation*}
$$

To compare equations (4) and (5) the logarithm of both sides of equations (4) and (5) were taken. The resulting equations, linear in $B, C, n, p$, and $q$, were fit to the available data using a least squares fit program. The program calculated the standard error,
$S_{Y, X}$, along with the vilues of $B, C, n, p$, and $q$. For this program
$S_{Y, X}=\sqrt{\sum_{i=1}^{M}\left[\ln \left(\frac{d a}{d N}\right)_{i_{\text {observed }}}-\ln \left(\frac{d a}{d N}\right)_{\left.i_{\text {calculated }}\right]^{2}}^{M}\right.}$
where $M$ is the number of data pairs used in the curve fit. It was decided that the standard error was the best means of comparing the two equations. The equation which consistently had tine smaller standard error would be the one which best fit the data. Since the standard error is being used as the measure of comparison, figures showing how equations (4) and (5) fit the experimental data were not prepared. The reader is referred to references (2). (3), and (6) for such figures.

Unfortunately or fortunately, depending on one's point of view, both equations gave almost identical results. The standard errors associated with fitting the values of $B$ and $C$ to equations (4) and (5) for the values of $K_{c}$ and $n$ recommended by Forman et $a l$ and the values of $p$ and $q$ recommended by Roberts and Erdogan are given in Tables 1 and 2. By comparing the values of $S_{Y, X}$ in these tables, it is seen that the two equations give comparable
results. It should be noted that the best fit values of $B, C, n, p$, and $q$, along with the associated standard error were calculated for equations (4) and (5). These results, although not included in this paper, indicate that the value of $n=3$ recommended by Forman et al, and $p=q=2$ recommended by Roberts and Erdogan were in reasonable agreement with the least squares fit of the data. The standard errors for this case again showed that the two equations give comparable results.

It is tine upinion of the present authors that the assumption made by Forman et al is correct in view of the data considered to date. The reasons for this are as follows:

1) The equation proposed gives excellent results when predicting the effect of mean stress on fatigue crack propagation due to plane extension for the two aluminum alloys considered.
2) The value of $K_{c}$ in the equation of Forman et al is in good agreement with values of $K_{c}$ obtained from fracture tests.

It is also the opinion of the present authors that the assumptions put forward by Roberts and Erdogan are correct. One reason for this is the ability of their
model to handle the effect of mean stress on fatigue crack propagation. A second and more important reason is the ability of their model to predict the rate of fatigue crack propagation due to bencling loads from data obtained in tensile tests.

By using the argument that fatigue crack propagation is more fundamentally related to plastic zone size and by observing the similarity of fracture modes between the cylindrical bending tests given in Appendix $A$ and results from plane extension, Roberts and Erdogan argued that for the same material if the plastic zone sizes for the two types of loading are the same, the growth rates should be the same. This led to the conclusion that the rate of fatigue crack propagation due to bending loads could be predicted from data obtained from tension testing. By modifying the stress-intensity factor for bending by a factor of $1 / 2$ and using the value of $C$ in equation (5) obtained from tensile data, one has

$$
\begin{equation*}
\frac{d a}{d N}=C(1+B)^{2}\left(\frac{\Delta K_{b}}{2}\right)^{4} \tag{7}
\end{equation*}
$$

where $\Delta K_{b}$ is the amplitude of the bending stress-intensity factor. The factor $1 / 2$ was determined theoretically so that the plastic zone sizes for both cases would be the same. Equation (7) was considered in reference (3).

Excellent results were obtained in predicting the fatigue crack propagation rates associated with the data given in Appendix A from fatigue crack propagation rates found in the literature for tensile data.

In a recent study, Catanach (ref. 10) investigated the rate of fatigue crack propagation in thin 6063-T6 aluminum shells. In his tests he subjected the shells which contained a longitudinal through crack to fluctuating internal pressure. As a result of the shell curvature the area near the crack tip is sinultaneously subjected to varying tensile and bending loads. Using the resuits of Roberts and Erdogan (refs. 3 and 6), it can be shown that the plast zone size can be approximated in the Dugdale sense (ref. ll) as being proportional to the square of $K_{t}+K_{b} / 2$, where $K_{t}$ is the stress-intensity factor for the in-plane tensile loads and $K_{b}$ is the stress-intensity factor for the bending loads. Catanach fit equation (5) to his data. He found that he got good results using a value of $\Delta K=\Delta K_{t}+\Delta K_{b} / 2$ in equation (5). Unfortunately the levels of $\nu_{\cdot b}$ compared to $K_{t}$ were so small that it was impossible to tell if the quantity $K_{t}+K_{b} / 2$ was the reason for the good fit.

Based on the previous thoughts concerning the models
of Forman et al and Roberts and Erdogan and the work or Catanach, an equation of the form

$$
\begin{equation*}
\frac{\mathrm{da}}{\mathrm{dN}}=D\left(\frac{K_{\max }}{K_{c}-K_{\max }}\right)^{r}(\Delta K)^{s} \tag{8}
\end{equation*}
$$

was fit to the data used to compare equations (4) and (5). The best fit values of $r$ and $s$ were $r \cong 1, s \cong 3$. Thus fixing the values of $r$ and $s$ at $l$ and 3 respectively, one arrives back at Forman's equation:

$$
\frac{d a}{d N}=\frac{B(1+\beta)(\Delta K)^{3}}{K_{c}-(1+\beta) \Delta K}
$$

The hypothesis of Roberts and Erdogan about the relationship between fatigue crack propagation rates due to tension and bending was tested by writing equation as

$$
\begin{equation*}
\frac{d a}{d N}=\frac{B(1+\beta)\left(\frac{\Delta K_{b}}{2}\right)^{3}}{K_{c}-(1+\beta)\left(\frac{\Delta K_{b}}{2}\right)} \tag{9}
\end{equation*}
$$

This equation was fit to the data found in Appendix $A$ for the 0.05 inch thick 2024-T3 aluminum for $\beta=0.392,0.632$, and 1.0. The value of $B$ found was $2.78 \times 10^{-12}$. This compares very well with the value of $B$ given in Table 3 for the 0.04 inch material from reference (5), $\Sigma=2.94 \times 10^{-12}$.

As a result of the above considerations, the following modification of Forman's equation is proposed:

$$
\begin{equation*}
\frac{d a}{d N}=\frac{c(1+\beta)\left(\Delta K_{e}\right)^{3}}{K_{c}-(1+\beta) \Delta K_{e}} \tag{10}
\end{equation*}
$$

where $K_{e}$ is defined as

$$
\begin{array}{ll}
K_{e}=K_{t} & \text {; for plane extension } \\
K_{e}=K_{b} / 2 & \text {; for bending } \\
K_{e}=K_{t}+K_{b} / 2 \text {; for combined loading }
\end{array}
$$

and $K_{t}$ and $K_{b}$ are the stress-intensity factors for extension and bending, respectively. This equation incorporates the concept of Forman et al that the fatigue crack propagation rate should become infinite as $K_{\text {max }}$ approaches $K_{c}$. It also incorporates the concept of Roberts and Eraogan that for similar fracture modes but different types of loading, the rate of fatigue crack propagation should be the same if the plastic zone sizes are the same.

With regard to the transverse bending tests, a suitable model for estimating the plastic zone size aue to the combined axial and transverse load has not been developed. When this is done a method for determining $K_{e}$ in equation (10) will be available.

1. The equations proposed by Forman et al and Roberts and Erdogan both handle the effect of mean stress on fatigue crack propagation equally well.
2. The following equation, a modification of Forman's equation, is proposed in place of the equations of Forman et al and Roberts and Erdogan:

$$
\frac{d a}{d N}=\frac{E(1+B)\left(\Delta K_{e}\right)^{3}}{K_{c}-(1+B) \Delta K_{e}}
$$

where $K_{e}$ is defined as
$K_{e}=K_{t} \quad$; for plane extension
$K_{e}=K_{b} / 2 \quad$; for bending $K_{e}=K_{t}+K_{b} / 2$; for combined loading
and $K_{t}$ and $K_{b}$ are the stress-intensity factors for extension and bending, respectively.

In closing, the authors would like to point out that the equation proposed was evaluated for only two aluninum alloys. Other materials might not show agreement with the equation. The hypothesis that $K_{e}=K_{t}+K_{b} / 2$
must be viewed with caution until it can be compared to more substantial data than found in reference (10). In general a large number of questions are not answered by equation (10) and should be the object of future studies.

## REFERENCES

1. P. C. Paris and F. Erdogan, "A Critical Analysis of Crack Propagation Laws," Journal of Basic Engineering, Trans. ASME, Series D, vol. 85, 1963, pp. 528-534.
2. R. G. Forman, V.E. Kearney, and R. M. Engle, "Numerical Analysis of Crack Propagation in Cyclic-Loaded Structures," Journal of Basic Engineering, Trans. ASME, Series D, vol. 89, 1967, pp. 459-463.
3. R. Roberts and F. Erdogan, "The Effect of Mean Stress on Fatigue Crack Propagation in Plates Under Extension and Bending," Journal of Basic Engineering, Trans. ASME, Series D, vol. 89, 1967, pp. 885-892.
4. D. R. Donaldson and W. E. Anderson, "Crack Propagation Behavior of Some Airframe Materials," Proceeding of the Crack Propagation Symposium, Cranfield, England, 1961.
5. Reports and Transactions, National Aero- and Astronautical Research Institute, Amsterdam, XXXI, 1965.
6. F. Erdogan and R. Roberts, "A Comparative Study of Crack Propagation in Plates Under Extension and Bending," Proceeding, International Conference on Fracture, Sendai, Japan, 1965.
7. W. Illg and A. J. McEvily, "The Rate of Fatigue Crack Propagation for Two Aluminum Alloys Under Completely Reversed Loading," NASATND-52, Oct. 1959.
8. J. R. Rice, Mechanics of Crack Tip Deformation and Extension by Fatigue," Fatigue Crack Propagation, ASTM STU 415, Am. Soc. Testing Mats., 1967, pp. 247-309.
9. C. M. Hudson and J. T. Scardina, "Effect of Stress Ratio on Fatigue Crack Growth in 7075-T6 Aluminum Alloy Sheet," presented at the National Symposium on Fracture Mechanics, Bethlehem, Pa., 1967.
10. W. M. Catanach, Jr., "A Study of Fatigue Crack Propagation in Thin Cylindrical Shells Containing a Meridioral Crack," A Ph.D. Thesis, Lehigh University, 1967.
11. D. S. Dugdale, "Yielding of Steel Sheets Containing Slits," Jour. Mech. Phys. Solids, vol. 8, 1960, pp. 100-104.
12. W. J. Valentine, "On the Rate of Crack Extension Due to Transverse Shear Fatigue Loading," M.S. Thesis, Lehigh University, 1965.

TABLE 1
Material Constant and Standard Error for the Equation of Roberts and Erdogan

$$
\frac{d a}{d N}=C(1+\beta)^{2} \Delta K^{4}
$$

| Material | Reference | Thickness | $C \times 10^{-21}$ | $S_{Y, X}$ |
| :---: | :---: | :---: | :---: | :---: |
| 2024-T3 | 5 | $\begin{aligned} & 0.080 \\ & 0.024 \\ & 0.040 \\ & 0.079 \\ & 0.118 \\ & 0.157 \\ & \hline \end{aligned}$ | $\begin{array}{r} 15.89 \\ 7.48 \\ 7.49 \\ 6.52 \\ 10.26 \\ 12.51 \end{array}$ | $\begin{aligned} & 0.6167 \\ & 0.7444 \\ & 0.4112 \\ & 0.5680 \\ & 0.3144 \\ & 0.3802 \end{aligned}$ |
|  | 4 | $\begin{aligned} & 0.020 \\ & 0.032 \\ & 0.040 \\ & 0.063 \\ & 0.081 \\ & 0.102 \end{aligned}$ | $\begin{array}{r} 6.92 \\ 8.12 \\ 5.73 \\ 7.49 \\ 4.26 \\ 10.19 \\ \hline \end{array}$ | $\begin{aligned} & 0.6153 \\ & 0.3266 \\ & 0.4017 \\ & 0.4807 \\ & 0.4892 \\ & 0.6469 \end{aligned}$ |
|  | 7 | 0.081 | 5.01 | 0.5503 |
| 7075-T6 | 5 | 0.080 | 82.27 | 0.4189 |
|  | 4 |  | $\begin{aligned} & 22.20 \\ & 23.81 \\ & 24.41 \end{aligned}$ | $\begin{aligned} & 0.4150 \\ & 0.1717 \\ & 0.5888 \end{aligned}$ |
|  | 7 | 0.081 | 5.31 | 0.5527 |
|  | 9 | 0.090 | 11.23 | 0.5818 |

## TABLE 2

Material Constant and Standard Error for
the Equation of Forman et al.

$$
\frac{d a}{d N}=\frac{B(1+\beta) \Delta K^{3}}{K_{c}-(1+B) \Delta K}
$$

$$
\begin{array}{lll}
2024-T 3 & : & K_{c}=47,000 \\
7075-T 6 & : & K_{C}=38,500
\end{array}
$$

| Material | Reference | Thickness | $C \times 10^{-12}$ | $S_{Y, X}$ |
| :---: | :---: | :---: | :---: | :---: |
| 2024-T3 | 5 | $\begin{aligned} & 0.080 \\ & 0.024 \\ & 0.040 \\ & 0.079 \\ & 0.118 \\ & 0.157 \end{aligned}$ | $\begin{aligned} & 2.69 \\ & 2.89 \\ & 2.94 \\ & 2.60 \\ & 4.09 \\ & 5.02 \end{aligned}$ | $\begin{aligned} & 0.3670 \\ & 0.7314 \\ & 0.4355 \\ & 0.4546 \\ & 0.2809 \\ & 0.4006 \end{aligned}$ |
|  | 4 | $\begin{aligned} & 0.020 \\ & 0.032 \\ & 0.040 \\ & 0.063 \\ & 0.081 \\ & 0.102 \end{aligned}$ | $\begin{aligned} & 2.55 \\ & 3.16 \\ & 2.20 \\ & 2.08 \\ & 1.07 \\ & 3.30 \end{aligned}$ | $\begin{aligned} & 0.5310 \\ & 0.4223 \\ & 0.2652 \\ & 0.2730 \\ & 0.2517 \\ & 0.5109 \end{aligned}$ |
|  | 7 | 0.081 | 1.53 | 0.5601 |
| 7075-T6 | 5 | 0.080 | 10.07 | 0.3489 |
|  | 4 | $\begin{aligned} & 0.064 \\ & 0.090 \\ & 0.102 \end{aligned}$ | $\begin{aligned} & 6.43 \\ & 7.06 \\ & 5.28 \end{aligned}$ | $\begin{aligned} & 0.3327 \\ & 0.2835 \\ & 0.3278 \end{aligned}$ |
|  | 7 | 0.081 | 1.22 | 0.3257 |
|  | 9 | 0.090 | 3.02 | 0.5466 |


| Plate <br> llumber | Unterial | Thickness $\left(i n_{0}\right)$ | $\begin{gathered} \sigma_{a} \\ (\mathrm{PSI}) \end{gathered}$ | $B=\left(\sigma_{m} / \sigma_{a}\right)$ |
| :---: | :---: | :---: | :---: | :---: |
| 1 | 2024-T3 Clad | 0.050 | 12,500 | 0 |
| 2 | ? 024 -13 Clad | 0.050 | 16,640 | 0 |
| 3 | 2024-13 Clad | 0.050 | 18,850 | 0 |
| 4 | 2024-93 Clad | 0.050 | 18,850 | 0 |
| 5 | 2024-T3 Clad | 0.050 | 18,850 | 0 |
| 6 | 2024-T3 Bare | 0.050 | 12,175 | 0.39 |
| 7 | 2024-T3 Bare | 0.050 | 12. 175 | 0.39 |
| 8 | 2024-T3 Bare | 0.050 | 12.475 | 0.68 |
| 9 | 2024-73 Bare | 0.050 | 12,435 | 0.62 |
| 10 | 2024-13 Bare | 0.050 | 8,320 | 1 |
| 11 | 2024-13 Bare | 0.050 | 8,320 | 1 |
| 12 | 2024-13 Bare | 0.080 | 21,800 | 0 |
| 13 | 2024-93 Bare | 0.080 | 26,000 | 0 |
| 14 | 2024-T3 Bare | 0.080 | 30,200 | 0 |
| 15 | 2024-13 Bare | 0.080 | 30,200 | 0 |
| 16 | 2024-13 Bare | 0.080 | 30,200 | 0 |
| 17 | 2024-13 Bare | 0.080 | 30,200 | 0 |
| 18 | 2024-T3 Clad | 0. 080 | 10,400 | 0 |
| 19 | 2024-13 Bare | 0.100 | 26,000 | 0 |
| 20 | 2024-93 Bare | 0.100 | 26,000 | 0 |
| 21 | 2024-T3 Bare | 0.100 | 26,000 | 0 |
| 22 | 2024-T3 Bare | 0.100 | 30.700 | 0 |
| 23 | 2024-T3 Bare | 0.100 | 35.400 | 0 |
| 24 | 2024-T3 Bare | 0.100 | 35,400 | 0 |
| 25 | 2024-T3 Clad | 0.100 | 13,000 | 0 |
| 26 | 2024-T3 Clad | 0.100 | 35,000 | 0 |
| 27 | 2024-73 Clad | 0.100 | 38,500 | 0 |
| 28 | 2024-13 Bare | 0.125 | 16,100 | 0 |
| 29 | 2024-13 Bare | 0.125 | 18,400 | 0 |
| 30 | 2024-T3 Bare | 0.125 | 18,400 | 0 |
| 31 | 2024-T3 Clad | 0.125 | 31.200 | 0 |
| 32 | 2024-T3 Clad | 0.125 | 37.400 | 0 |
| 33 | 2024-T3 Clad | 0.125 | 44,700 | 0 |
| 34 | 2024-T3 Clad | 0.160 | 19.750 | 0 |
| 35 | 2024-43 Clad | 0.160 | 19.750 | 0 |
| 36 | 2024-T3 Clad | 0.160 | 19.750 | 0 |
| 37 | 2024-93 clad | 0.160 | 19.750 | 0 |
| 38 | 2024-93 Clad | 0.160 | 23,100 | 0 |
| 39 | 2024-53 Clad | 0.160 | 29,600 | 0 |
| 40 | 7075-T6 Clad | 0.050 | 21,800 | 0 |
| 41 | 7075-T6 Clad | 0.050 | 21,800 | 0 |
| 42 | 7075-T6 Clad | 0.050 | 29,800 | 0 |
| 43 | 7075-T6 clad | 0.100 | 28,000 | 0 |
| 44 | 7075-16 Clad | 0.100 | 28,000 | 0 |
| 45 | 7075-T6 Clad | 0.100 | 28,000 | 0 |
| 46 | 7075-76 Clad | 0.100 | 34,400 | 0 |
| 47 | 7075-56 clad | 0.100 | 34,400 | 0 |
| 48 | 7075-T6 Clad | 0.100 | 44, 200 | 0 |
| 49 | 7075-T6 Clad | 0.100 | 44,200 | 0 |

APPMDIX A (cont.)

| Plate <br> Number | Material | Thicloness (1n.) | $\sigma_{(P S I)}^{a}$ | $\beta=\left(\sigma_{m} / \sigma_{z}\right)$ |
| :---: | :---: | :---: | :---: | :---: |
| 50 | 7075-96 Clad | 0.100 | 52,500 | 0 |
| 51 | 7075-76 Bare | 0.120 | 14,80C | 0 |
| 52 | 7075-T6 Bare | 0.120 | 31,200 | 0 |
| 53 | 7075-76 Bare | 0.120 | 34,800 | 0 |
| 54 | 7075-T6 Bare | 0.120 | 42,500 | 0 |

plate

| $\frac{20}{0.687}$ | $\frac{\text { El } 10^{3}}{325}$ |
| :--- | :--- |
| 0.704 | 340 |
| 0.713 | 365 |
| 0.755 | 395 |
| 0.814 | 419 |
| 0.878 | 430 |
| 0.900 | 445 |
| 0.974 | 460 |
| 1.007 | 475 |
| 1.069 | 490 |
| 1.083 | 595 |
| 1.154 | 520 |
| 1.226 | 535 |
| 1.244 | 550 |
| 1.298 | 565 |
| 1.570 | 580 |
| 1.641 | 585 |

plate 42

| 2. | 103 |
| :---: | :---: |
| 0.328 | 55 |
| 0.353 | 65 |
| 0.407 | 75 |
| 0.423 | 85 |
| 0.460 | 95 |
| 0.510 | 105 |
| 0.549 | 117 |
| 0.597 | 130 |
| 0.638 | 145 |
| 0.793 | 160 |
| 0.855 | 170 |
| 0.883 | 180 |
| 0.984 | 190 |
| 1.067 | 200 |
| 1.141 | 210 |
| 1.225 | 215 |
| 1.465 | 220 |
| 1.563 | 225 |
| 1.782 | 230 |

plate 13

|  | $2 n$ |
| :---: | :---: |
| 0.361 | 50 |
| 0.010 | 60 |
| 0.486 | 70 |
| 0.502 | 80 |

Prate 3 cont.

| $\frac{2 a}{0.553}$ | 91 |
| :---: | :---: |
| 0.600 | 100 |
| 0.659 | 110 |
| 0.711 | 120 |
| 0.854 | 130 |
| 1.047 | 135 |
| 1.107 | 137 |
| 1.247 | 140 |
| 1.319 | 142 |
| 1.330 | 143 |
| 1.431 | 145 |
| 1.589 | 147 |
| 1.664 | 148 |

Plate 4

| $\frac{28}{0.271}$ | $\frac{7 \times 10^{3}}{25}$ |
| :--- | :--- |
| 0.288 | 35 |
| 0.302 | 45 |
| 0.368 | 60 |
| 0.413 | 70 |
| 0.451 | 80 |
| 0.485 | 90 |
| 0.541 | 100 |
| 0.594 | 110 |
| 0.710 | 120 |
| 0.937 | 130 |
| 1.053 | 133 |
| 1.096 | 135 |
| 1.120 | 140 |
| $\therefore .476$ | 145 |
| 1.569 | .45 |
| 1.768 | .49 |

Plate 15

| 20 | $\underline{\underline{10} 10^{3}}$ |
| :---: | :---: |
| 0.361 | 50 |
| 0.401 | 60 |
| 0.486 | 70 |
| 0.502 | 80 |
| 0.553 | 91 |
| 0.600 | 100 |
| 0.659 | 110 |
| 0.711 | 120 |
| 0.854 | 130 |
| 1.047 | 135 |

Plate 15 cont.

| $\frac{2 a}{1.107}$ | $\frac{137}{13}$ |
| :--- | :--- |
| 1.247 | 140 |
| 1.319 | 142 |
| 1.330 | 143 |
| 1.431 | 145 |
| 1.589 | 147 |
| 1.664 | 148 |

Plate 46

| 2 a | $\mathrm{Kx}_{\underline{1}} 10^{3}$ | 0.452 | 178 |
| :---: | :---: | :---: | :---: |
| 0.238 | 0 | 0.511 | 224 |
| 0.241 | 6 | 0.542 | 243 |
| 0.243 | 12 | 0.579 | 262 |
| 0.256 | 36 | 0.619 | 282 |
| 0.278 | 59 | 0.649 | 298 |
| 0.281 | 77 | 0.695 | 316 |
| 0.281 | 95 | 0.748 | 332 |
| 0.304 | 116 | 0.807 | 347 |
| 0.319 | 140 | 0.899 | 364 |
| 0.336 | 160 | 0.952 | 374 |
| 0.350 | 178 | 1.060 | 389 |
| 0.377 | 200 | 1.195 | 405 |
| 0.403 | 224 | 1.328 | 417 |
| 0.419 | 243 | 1.497 | 429 |
| 0.442 | 262 | 1.552 | 433 |

Plate +8

| $\frac{28}{0.282}$ |  |
| :--- | :--- |
| 0.155 |  |
| 0.296 | 175 |
| 0.312 | 190 |
| 0.340 | 205 |
| 0.368 | 218 |
| 0.393 | 230 |
| 0.419 | 240 |
| 0.449 | 250 |
| 0.478 | 260 |
| 0.505 | 270 |
| 0.538 | 280 |
| 0.578 | 290 |
| 0.630 | 300 |
| 0.683 | 310 |
| 0.748 | 320 |
| 0.827 | 330 |
| 0.922 | 340 |
| 1.039 | 350 |

Plate 8 cont.

| 2 a | $\mathrm{N} \times 10^{3}$ |
| :---: | :---: |
| 1.182 | 360 |
| 1.329 | 369 |
| 1.460 | 374 |
| 1.704 | 384 |
| Plate 49 |  |
| 2a | $\underline{12 x 10^{3}}$ |
| 0.303 | 90 |
| 0.363 | 125 |
| 0.436 | 155 |
| 0.495 | 175 |
| 0.549 | 190 |
| 0.619 | 205 |
| 0.697 | 218 |
| 0.784 | 230 |
| 0.873 | 240 |
| 0.921 | 245 |
| 0.978 | 250 |
| 1.031 | 255 |
| 1.099 | 260 |
| 1.174 | 265 |
| 1.222 | 268 |
| 1.254 | 270 |
| 1.301 | 272 |
| 1.348 | 275 |
| 1.458 | 280 |
| 1.538 | 284 |
| 1.632 | 287 |
| 1.716 | 290 |

Plate $\$ 10$

| 2 a | $\underline{\mathrm{I}} \times 10^{3}$ |
| :---: | :---: |
| 0.242 | 200 |
| 0.358 | 514 |
| 0.419 | 600 |
| 0.566 | 650 |
| 0.600 | 780 |
| 0.694 | 790 |
| 0.819 | 810 |
| 0.886 | 835 |
| 0.951 | 860 |
| 1. 048 | 880 |
| 1.134 | 895 |
| 1.232 | 915 |
| 1.421 | 927 |
| 1.532 | 941 |
| 1.823 | 955 |

Plate R11

| 2a | $\underline{\mathrm{I}} \times 10^{3}$ |
| :---: | :---: |
| 0.232 | 0 |
| 0.245 | 90 |
| 0.267 | 200 |
| 0.354 | 342 |
| 0.432 | 514 |
| 0.518 | 600 |
| 0.546 | 650 |
| 0.642 | 730 |
| 0.711 | 790 |
| 0.741 | 810 |
| 0.835 | 835 |
| 0.886 | 860 |
| 0.889 | 880 |
| 0.936 | 895 |
| 1.021 | 915 |
| 1.054 | 927 |
| 1.121 | 941 |
| 1.166 | 955 |
| 1.265 | 970 |
| 1.362 | 986 |
| 1.476 | 1000 |
| 1.566 | 1020 |

Plate $\ddagger 12$

| 2 a | $\mathrm{H} \times 10^{3}$ |
| :---: | :---: |
| 0.299 | 5 |
| 0.326 | 10 |
| 0.356 | 15 |
| 0.424 | 25 |
| 0.518 | 35 |
| 0.648 | 45 |
| 0.841 | 55 |
| 1.155 | 65 |
| 1.867 | 76 |

Plate 113

|  |  |
| :--- | ---: |
| $\frac{2 \pi}{0.238}$ |  |
| 0.270 |  |
| 0.279 | 1 |
| 0.318 | 3 |
| 0.350 | 5 |
| 0.382 | 7 |
| 0.436 | 10 |
| 0.504 | 13 |
| 0.581 | 16 |
| 0.683 | 19 |
| 0.823 | 22 |

Plate \#13 cont.

| $\frac{2 \mathrm{a}}{1.032}$ | $\frac{\mathrm{~K} \times 10^{3}}{25}$ |
| :--- | ---: |
| 1.432 | 28 |
| 1.746 | 29 |

Plate 14

|  |  | 28 | $\underline{\mathrm{Na}} 0^{3}$ |
| :---: | :---: | :---: | :---: |
| 2 a | $\mathrm{NrI} 10^{3}$ | $\overline{0.280}$ | 1 |
| 0.299 | 1 | 0.319 | 2.25 |
| 0.366 | 3 | 0.341 | 3 |
| 0.432 | 5 | 0.366 | 4 |
| 0.509 | 7 | 0.434 | 6 |
| 0.604 | 9 | 0.528 | 8 |
| 0.720 | 11 | 0.658 | 10 |
| 0.904 | 13 | 1.048 | 13 |
| 1.317 | 15 | 1.428 | 14 |
| 1.580 | 15.8 | 1.674 | 14.3 |
| 1.744 | 16.25 |  |  |

Plate $\$ 18$
Plate 15

| 2 a | $\underline{\mathrm{K} \times 10^{3}}$ |
| :---: | :---: |
| 0.231 | 0 |
| 0.261 | 1 |
| 0.287 | 2 |
| 0.346 | 4 |
| 0.385 | 6 |
| 0.441 | 8 |
| 0.535 | 11 |
| 0.689 | 15 |
| 0.794 | 17 |
| 0.927 | 19 |
| 1.096 | 21 |
| 1.223 | 22 |
| 1.302 | 22.5 |
| 1.527 | 23.5 |


| 2 a | 판 ${ }^{3}$ |
| :---: | :---: |
| 0.271 | 700 |
| 0.294 | 850 |
| 0.302 | 878 |
| 0.310 | 900 |
| 0.320 | 927 |
| 0.330 | 953 |
| 0.341 | 986 |
| 0.347 | 1,000 |
| 0.367 | 1,049 |
| 0.395 | 1,113 |
| 0.438 | 1,186 |
| 0.452 | 1,215 |
| 0.489 | 1,269 |
| 0.519 | 1,317 |
| 0.570 | 1,394 |
| 0.668 | 1,497 |
| 0.765 | 1.575 |
| 0.836 | 1,631 |
| 0.879 | 1,661 |
| 0.978 | 1.731 |
| 1.081 | 1,800 |
| 1.222 | 1,869 |
| 1.323 | 1,910 |
| 1.444 | 1.950 |
| 1.480 | 1,960 |
| 1.523 | 1,971 |
| 1.560 | 1,980 |

Plate \#19

| $\frac{2 a}{2 a}$ | $=10^{3}$ |
| :--- | ---: |
|  | 2 |
| 0.346 | 5 |
| 0.363 | 6 |
| 0.382 | 7 |
| 0.401 | 8 |
| 0.422 | 9 |
| 0.439 | 10 |
| 0.488 | 12 |
| 0.532 | 14 |
| 0.593 | 15 |
| 0.654 | 18 |
| 0.719 | 20 |
| 0.792 | 22 |
| 0.889 | 24 |
| 0.982 | 26 |
| 1.033 | 27 |
| 1.096 | 28 |
| 1.161 | 29 |
| 1.235 | 30 |
| 1.316 | 31 |
| 1.408 | 32 |
| 1.516 | 33 |
| 1.647 | 34 |
| 1.798 | 35 |

Plate \#20

| 2 a | 18x10 ${ }^{3}$ |
| :---: | :---: |
| 0.850 | 2.0 |
| 0.928 | 3.0 |
| 1.011 | 4.0 |
| 1.056 | 4.5 |
| 1.105 | 5.0 |
| 1.158 | 5.5 |
| 1.208 | 6.0 |
| 1.264 | 6.5 |
| 1.321 | 7.0 |
| 1.378 | 7.5 |
| 1.442 | 8.0 |
| 1.503 | 8.5 |
| 1.564 | 9.0 |
| 1.621 | 9.5 |
| 1.682 | 10.0 |
| 1.737 | 10.5 |

Plate \#21

| 2a | N(1010 ${ }^{3}$ |
| :---: | :---: |
| 0.256 | 0.5 |
| 0.266 | 1.0 |
| 0.278 | 1.5 |
| 0.290 | 2.0 |
| 0.310 | 3.0 |
| 0.328 | 4 |
| 0.345 | 5 |
| 0.361 | 6 |
| 0.378 | 7 |
| 0.396 | 8 |
| 0.419 | 9 |
| 0.436 | 10 |
| 0.464 | 11 |
| 0.483 | 12 |
| 0.508 | 13 |
| 0.532 | 14 |
| 0.560 | 15 |
| 0.587 | 16 |
| 0.618 | 17 |
| 0.652 | 18 |
| 0.688 | 19 |
| 0.735 | 20 |
| 0.767 | 21 |
| 0.806 | 22 |
| 0.852 | 23 |
| 0.900 | 24 |
| 0.950 | 25 |
| 0.999 | 26 |
| 1.059 | 27 |
| 1.121 | 28 |
| 1.296 | 29 |
| 1.272 | 30 |
| 1.370 | 31 |
| 1.478 | 32 |
| 1.593 | 33 |
| 1.766 | 34 |


| $\frac{2 a}{0.322}$ |  |
| :--- | :--- |
| $0.3 x 10^{3}$ |  |
| 0.50 |  |
| 0.410 | 1.00 |
| 6.457 | 2.00 |
| 0.508 | 2.50 |
| 0.562 | 3.00 |
| 0.637 | 3.50 |
| 0.702 | 4.00 |
| 0.796 | 4.50 |
| 0.853 | 4.75 |
| 0.913 | 5.00 |
| 0.984 | 5.25 |
| 1.082 | 5.50 |
| 1.254 | 5.80 |
| 1.451 | 6.00 |

Plate +25

| 2 a | K×10 ${ }^{3}$ |
| :---: | :---: |
| 0.283 | 253 |
| 0.297 | 277 |
| 0.303 | 300 |
| 0.346 | 400 |
| 0.398 | 500 |

Plate 击22 cont

Plate \#25 cont.

| $\frac{2 \mathrm{a}}{0}$ | $\mathrm{~N} \times 10^{3}$ |
| :--- | ---: |
| 0.454 | 600 |
| 0.530 | 703 |
| 0.569 | 750 |
| 0.621 | 800 |
| 0.744 | 905 |
| 0.397 | 995 |
| 1.046 | 1,052 |
| 1.117 | 1,071 |
| 1.228 | 1,100 |
| 1.336 | 1,120 |
| 1.410 | 1,135 |
| 1.477 | 1,146 |

Plate \#26

| $\frac{2 \mathrm{a}}{0.400}$ | $\frac{\mathrm{~N} \times 10^{3}}{2}$ |
| :--- | :--- |
| 0.583 | 4 |
| 1.035 | 6 |
| 1.706 | 6.5 |

Plate \#27

| $\frac{2 a}{}$ | $\times 10^{3}$ |
| :--- | :--- |
| 0.318 | 1.0 |
| 0.401 | 2.0 |
| 0.459 | 3.0 |
| 0.624 | 4.5 |
| 0.802 | 5.5 |
| 0.978 | 6.2 |
| 1.067 | 6.4 |
| 1.193 | 6.6 |
| 1.343 | 6.8 |
| 1.633 | 7.0 |

Plate \#28

| 20 | $\mathrm{HK} \times 10^{3}$ |
| :---: | :---: |
| 0.298 | 82 |
| 0.323 | 98 |
| 0.338 | 107 |
| 0.361 | 122 |
| 0.406 | 145 |
| 0.434 | 164 |
| 0.457 | 169 |
| 0.491 | 188 |
| 0.512 | 197 |
| 0.551 | 214 |

Plate \#28 cont.

| $\frac{2 a}{2 a}$ |  |
| :--- | :--- |
| 0.587 |  |
| 0.626 | 226 |
| 0.700 | 261 |
| 0.700 | 275 |
| 0.767 | 275 |
| 0.811 | 285 |
| 0.863 | 294 |
| 0.932 | 306 |
| 1.004 | 316 |
| 1.073 | 324 |
| 1.127 | 331 |
| 1.248 | 334 |
| 1.338 | 352 |
| 1.431 | 361 |
| 1.538 | 370 |
| 1.606 | 375 |
| 1.675 | 380 |


| Plate \# 29 |  |
| :---: | :---: |
| 2 a | W×10 ${ }^{3}$ |
| $\overline{0.246}$ | 6 |
| 0.269 | 15 |
| 0.289 | 20 |
| 0.305 | 29 |
| 0.331 | 35 |
| 0.346 | 40 |
| 0.386 | 50 |
| 0.414 | 61 |
| 0.492 | 75 |
| 0.569 | 84 |
| 0.710 | 96 |
| 0.810 | 102 |
| 0.873 | 105 |
| 1.075 | 113 |
| 1.144 | 115 |
| 1.303 | 120 |
| 1.370 | 121 |
| 1.459 | 123 |
| 1.520 | 125 |

Plate +30

| 22 | $\underline{\mathrm{H} \times 10^{3}}$ |
| :---: | :---: |
| 0.258 | 35 |
| 0.304 | 47 |
| 0.334 | 56 |
| 0.359 | 64 |
| 0.411 | 78 |

Plate \#30 cont.

| $\frac{2 a}{0.445}$ | $\frac{\mathrm{~N} \times 10^{3}}{88}$ |
| :--- | ---: |
| 0.500 | 100 |
| 0.600 | 115 |
| 0.649 | 121 |
| 0.749 | 131 |
| 0.811 | 136 |
| 0.883 | 141 |
| 0.988 | 147 |
| 1.156 | 154 |
| 1.272 | 159 |
| 1.365 | 162 |
| 1.462 | 165 |
| 1.533 | 167 |
| 1.610 | 169 |

Plate +31

| $\frac{2 a}{0.291}$ | $\frac{N \times 10^{3}}{1}$ |
| :--- | :---: |
| 0.338 | 2 |
| 0.376 | 3 |
| 0.419 | 4 |
| 0.461 | 5 |
| 0.508 | 6 |
| 0.560 | 7 |
| 0.653 | 8 |
| 0.698 | 9 |
| 0.783 | 10 |
| 0.897 | 11 |
| 1.025 | 12 |
| 1.269 | 13.2 |
| 1.360 | 13.5 |
| 1.482 | 13.8 |
| 1.587 | 14.0 |
| 1.734 | 14.2 |

Plate +32

| $\frac{2 a}{0.341}$ | Kri0 |
| :--- | :--- |
| 0.3 | 0.5 |
| 0.4444 | 1.2 |
| 0.490 | 1.5 |
| 0.587 | 2.0 |
| 0.727 | 2.5 |
| 0.999 | 3.0 |
| 1.302 | 3.2 |
| 1.583 | 3.3 |
| 1.843 | 3.35 |

Plate \#33
$\begin{array}{ll}2 \mathrm{a} & \mathrm{NXI} 0^{3} \\ 0.297 & 0.1 \\ 0.347 & 0.2 \\ 0.422 & 0.4 \\ 0.495 & 0.6 \\ 0.581 & 0.8 \\ 0.729 & 1.0 \\ 0.856 & 1.1 \\ 1.073 & 1.2 \\ 2.029 & 1.3\end{array}$

## Plate $\# 34$

$\frac{2 a}{0.345} \frac{N\left(10^{3}\right.}{56}$
$\begin{array}{ll}0.385 & 58 \\ 0.394 & 60\end{array}$
$\begin{array}{ll}0.105 & 66 \\ 0.436 & 74\end{array}$
$0.460 \quad 80$
$\begin{array}{ll}0.530 & 88 \\ 0.580 & 94\end{array}$
$0.644 \quad 98$
$0.688 \quad 104$
$0.744 \quad 108$
$0.791 \quad 114$
0.857122
$\begin{array}{ll}0.920 & 125 \\ 0.990 & 130\end{array}$
$\begin{array}{ll}0.990 & 130 \\ 1.031 & 135\end{array}$
$\begin{array}{ll}1.087 & 136 \\ 1.133 & 137\end{array}$
$1.190 \quad 146$
$\begin{array}{ll}1.256 & 150 \\ 1.288 & 152\end{array}$
$1.314 \quad 154$
$1.421 \quad 155$
1.461157
1.479158
1.517160

Plate 435

|  | - $10^{3}$ | Plate 37 |  |
| :---: | :---: | :---: | :---: |
| $\frac{28}{0.257}$ | $\frac{1800}{7}$ | 2a |  |
| 0.260 | 20 | 0.241 | 35 |
| 0.280 | 30 | 0.317 | 50 |
| 0.341 | 40 | 0.379 | 60 |
| 0.394 | 48 | 0.416 | 70 |

Plate 22 cont. Plate \#40

| $\frac{2 a}{0.477}$ | $\frac{N 10}{} 80$ |
| :--- | ---: |
| 0.512 | 90 |
| $\therefore .602$ | 100 |
| 1.448 | 105 |
| 0.103 | 110 |
| 0.768 | 116 |
| 0.819 | 123 |
| 0.951 | 131 |
| 1.046 | 138 |
| 1.104 | 141 |
| 1.227 | 146 |
| 1.329 | 152 |
| 1.388 | 156 |
| 1.520 | 161 |

Plate \# 38

| 2 a | $\underline{\mathrm{Fr}} 10^{3}$ |
| :---: | :---: |
| 0.299 | 6 |
| 0.356 | 10 |
| 0.435 | 16 |
| 0.487 | 22 |
| 0.552 | 28 |
| 0.626 | 34 |
| 0.729 | 40 |
| 0.796 | 44 |
| 0.893 | 49 |
| 1.015 | 54 |
| 1.118 | 58 |
| 1.245 | 62 |
| 1.369 | 66 |
| 1.595 | 71 |

Plate 439

| 2 a | $\mathrm{N} \times 10^{3}$ | Plate 442 |  |
| :---: | :---: | :---: | :---: |
| 0.367 | 6.0 |  |  |
| 0.486 | 9.0 |  |  |
| 0.605 | 13.0 | $2 a$ | $\mathrm{Nr}^{10}{ }^{3}$ |
| 0.700 | 15.5 | 0.234 | 3 |
| 0.792 | 17.5 | 0.255 | 6 |
| 0.930 | 20.0 | 0.294 | 9 |
| 1.067 | 22.5 | 0.352 | 12 |
| 1.167 | 24.0 | 0.425 | 15 |
| 1.246 | 25.0 | 0.494 | 18 |
| 1.335 | 26.0 | 0.585 | 21 |
| 1.438 | 27.2 | 0.668 | 24 |
| 1.530 | 28.5 | 0.936 | 27 |


| $2 a$ | $\frac{N 10}{}{ }^{3}$ |
| :--- | ---: |
| 0.266 | 13 |
| 0.302 | 20 |
| 0.328 | 27 |
| 0.397 | 34 |
| 0.452 | 41 |
| 0.528 | 48 |
| 0.633 | 54 |
| 0.717 | 60 |
| 0.789 | 65 |
| 0.883 | 70 |
| 1.115 | 76 |
| 1.172 | 78 |
| 1.266 | 80 |
| 1.416 | 83 |
| 1.625 | 86 |


| Plate \#41 |  |
| :--- | ---: |
|  |  |
| $\frac{2 a}{2 a}$ | सx10 |
| 0.219 | 0 |
| 0.338 | 35 |
| 0.360 | 40 |
| 0.415 | 45 |
| 0.443 | 50 |
| 0.470 | 55 |
| 0.577 | 65 |
| 0.642 | 70 |
| 0.685 | 75 |
| 0.756 | 80 |
| 0.835 | 85 |
| 0.939 | 90 |
| 1.053 | 95 |
| 1.245 | 100 |
| 1.499 | 103 |
| 1.653 | 105 |

Plate 442

Plate\#42 cont.

| $\frac{2 a}{1.178}$ | $\frac{\mathrm{~N} \times 10^{3}}{30}$ |
| :--- | ---: |
| 1.530 | 32 |
| 1.774 | 33 |

Plate \#43
$\begin{array}{lr}\frac{2 a}{0.390} & \frac{\mathrm{~N} \times 10^{3}}{10} \\ 0.441 & 11 \\ 0.482 & 12 \\ 0.521 & 13 \\ 0.558 & 14 \\ 0.636 & 15 \\ 0.687 & 16 \\ 0.769 & 17 \\ 0.829 & 18 \\ 0.389 & 19 \\ 0.954 & 20 \\ 1.009 & 21 \\ 1.099 & 22 \\ 1.231 & 23 \\ 1.353 & 24 \\ 1.550 & 25 \\ 1.895 & 26\end{array}$

Plate \#44 cont.

$$
\begin{array}{ll}
\frac{2 a}{0.663} & \frac{N X 10^{3}}{20.0} \\
0.716 & 21.0 \\
0.876 & 22.0 \\
0.913 & 22.5 \\
0.971 & 23.0 \\
1.005 & 23.5 \\
1.089 & 24.0 \\
1.252 & 24.5 \\
1.214 & 25.0 \\
1.288 & 25.5 \\
1.391 & 26.0 \\
1.508 & 26.5 \\
1.631 & 27.0 \\
1.818 & 27.5 \\
2.062 & 28.0
\end{array}
$$

## Plate $\# 45$

| 2a | $\frac{\mathrm{Nax} 10^{3}}{3}$ |
| :---: | :---: |
| 0.247 | 30.0 |
| 0.234 | 30.5 |
| 0.307 | 31.0 |
| 0.322 | 31.5 |
| 0.327 | 33.0 |
| 0.354 | 33.5 |
| 0.362 | 34.0 |
| 0.364 | 34.5 |
| 0.388 | 35.0 |
| 0.395 | 35.5 |
| 0.405 | 36.0 |
| 0.410 | 36.5 |
| 0.422 | 37.0 |
| 0.434 | 38.0 |
| 0.470 | 39.0 |
| 0.480 | 39.5 |
| 0.493 | 40.0 |
| 0.514 | 40.5 |
| 0.527 | 41.0 |
| 0.540 | 41.5 |
| 0.549 | 42.0 |
| 0.572 | 42.5 |
| 0.600 | 43.5 |
| 0.620 | 44.0 |
| 0.660 | 45.0 |
| 0.690 | 46.0 |
| 0.725 | 46.5 |
| 0.751 | 47.0 |
| 0.772 | 47.5 |
| 0.814 | 48.0 |

Plate 445 cont.

| 3 a | Nx10 |
| :---: | :---: |
| 0.834 | 48.5 |
| 0.846 | 49. |
| 0.883 | 49 |
| 0.906 | 50.0 |
| 0.969 | 50.5 |
| 1.007 | 51. |
| 1.097 | 52.0 |
| 1.178 | 52. |
| 1.221 | 53.0 |
| 1.292 | 53.5 |
| 1.379 | 54.0 |
| 1.498 | 54.5 |
| 1.730 | 55.0 |
| 2.025 | 55.5 |

## Plate \#46

| 2a | $\underline{\mathrm{F} \times 10^{3}}$ |
| :---: | :---: |
| 0.195 | 0.0 |
| 0.216 | 0.5 |
| 0.233 | 1.0 |
| 0.262 | 1.5 |
| 0.277 | 2.0 |
| 0.305 | 2.5 |
| 0.327 | 3.0 |
| 0.351 | 3.5 |
| 0.368 | 4.0 |
| 0.395 | 4.5 |
| 0.428 | 5.0 |
| 0.451 | 5.5 |
| 0.466 | 6.0 |
| 0.490 | 6.5 |
| 0.525 | 7.0 |
| 0.552 | 7.5 |
| 0.591 | 8.0 |
| 0.613 | 8.5 |
| 0.645 | 9.0 |
| 0.688 | 9.5 |
| 0.732 | 10.0 |
| 0.759 | 10.5 |
| 0.819 | 11.0 |
| 0.895 | 11.5 |
| 0.932 | 12.0 |
| 1.002 | 12.5 |
| 1.052 | 13.0 |
| 1.133 | 13.5 |
| 1.196 | 14.0 |
| 1.278 | 14.5 |
| 1.406 | 15.0 |
| 1.611 | 15.5 |

Plate \#46 cont.
$\frac{2 a}{2.067} \frac{\mathrm{NxI} 0^{3}}{16.0}$

Plate \#47
$\begin{array}{lllll} & & 0.325 & 1.0 \\ \frac{2 a}{0} & & 0.382 & 1.5 \\ 0.222 & 0.0 & 0.427 & 2.0 \\ 0.300 & 2.5 & 0.453 & 2.2 \\ 0.321 & 3.0 & 0.523 & 2.6 \\ 0.339 & 3.5 & 0.666 & 3.0 \\ 0.362 & 4.0 & 0.723 & 3.1 \\ 0.376 & 4.5 & 0.769 & 3.2 \\ 0.403 & 5.0 & 0.845 & 3.4 \\ 0.434 & 5.5 & 0.930 & 3.5 \\ 0.458 & 6.0 & 1.037 & 3.55 \\ 0.486 & 6.5 & & \\ 0.511 & 7.0 & & \\ 0.550 & 7.5 & \text { Plate }+50\end{array}$

Plate 448

| 2 a | $\mathrm{Nx} 10^{3}$ | Prate 451 |  |
| :---: | :---: | :---: | :---: |
| 0.261 | 0.50 |  |  |
| 0.306 | 1.0 | 2 a | $\underline{1-10}$ |
| 0.329 | 1.3 | 0.273 | 800 |
| 0.345 | 1.5 | 0.335 | 830 |
| 0.380 | 1.8 | 0.391 | 855 |
| 0.408 | 2.0 | 0.443 | 875 |
| 0.439 | 2.2 | 0.527 | 905 |
| 0.472 | 2.4 | 0.656 | 930 |
| 0.518 | 2.6 | 0.799 | 950 |
| 0.561 | 2.8 | 1.030 | 970 |
| 0.633 | 3.0 | 1.202 | 980 |
| 0.758 | 3.2 | 1.308 | 985 |
| 0.958 | 3.4 | 1.429 | 990 |
| 1.742 | 3.5 | 1.567 | 995 |


| Plate 449 |  | Plate \#52 |  |
| :---: | :---: | :---: | :---: |
| 2a | $\mathrm{N} \times 10^{3}$ | 28 | $\underline{\mathrm{Na}} 10^{3}$ |
| 0.242 | 0.2 | 0.261 | 1 |
| 0.273 | 0.5 | 0.291 | 2 |
| 0.305 | 0.8 | 0.320 | 3 |
| 0.325 | 1.0 | 0.355 | 4 |
| 0.382 | 1.5 | 0.383 | 5 |
| 0.427 | 2.0 | 0.416 | 6 |
| 0.453 | 2.2 | 0.450 | 7 |
| 0.523 | 2.6 | 0.495 | 8 |
| 0.635 | 3.0 | 0.530 | 9 |
| 0.666 | 3.1 | 0.576 | 10 |
| 0.723 | 3.2 | 0.622 | 11 |
| 0.769 | 3.3 | 0.679 | 12 |
| 0.845 | 3.4 | 0.734 | 13 |
| 0.930 | 3.5 | 0.808 | 14 |
| 1.037 | 3.55 | $0.88 \%$ | 15 |
|  |  | 0.967 | 16 |
|  |  | 1.061 | 17 |
| Plate 450 |  | 1.176 | 18 |
|  |  | 1.240 | 18.5 |
| 2 a | $\mathrm{NXIO}^{3}$ | 1.312 | 19 |
| 0.281 | 0.2 | 1.390 | 19.5 |
| 0.298 | 0.3 | 1.468 | 20 |
| 0.337 | 0.4 | 1.583 | 20.5 |
| 0.351 | 0.5 | 1.707 | 21 |
| 0.385 | 0.6 |  |  |
| 0.411 | 0.7 |  |  |
| 0.464 | 0.85 | Plate |  |
| 0.534 | 0.9 |  |  |
| 0.635 | 1.0 | 2 a | $\underline{16 \times 10^{3}}$ |
| 0.918 | 1.1 | 0.243 | 1 |
| 2.230 | 1.15 | 0.275 | 2 |
|  |  | 0.309 | 3 |
|  |  | 0.343 | 4 |
| Plate 451 |  | 0.387 | 5 |
|  |  | 0.423 | 6 |
| 2 a | $\underline{\mathrm{Br}} 10^{3}$ | 0.486 | 7 |
| 0.273 | 800 | 0.540 | 8 |
| 0.335 | 830 | 0.599 | 9 |
| 0.391 | 855 | 0.670 | 10 |
| 0.443 | 875 | 0.756 | 11 |
| 0.527 | 905 | 0.851 | 12 |
| 0.656 | 930 | 0.962 | 13 |
| 0.799 | 950 | 1.028 | 13.5 |
| 1.030 | 970 | 1.101 | 14 |
| 1.202 | 980 | 1.180 | 14.5 |
| 1.308 | 985 | 1.266 | 15 |
| 1.429 | 990 | 1.366 | 15.5 |
| 1.567 | 995 | 1.476 | 16 |
|  |  | 1.551 | 16.3 |
|  |  | 1.617 | 16.5 |
|  |  | 1.728 | 16.8 |

## Plate \#54

| $\frac{2 a}{0.274}$ | $\frac{\mathrm{~N} \times 10^{3}}{0.5}$ |
| :--- | ---: |
| 0.319 | 1.0 |
| 0.362 | 1.5 |
| 0.408 | 2.0 |
| 0.459 | 2.5 |
| 0.515 | 3.0 |
| 0.596 | 3.5 |
| 0.679 | 4.0 |
| 0.794 | 4.5 |
| 0.963 | 5.0 |
| 1.058 | 5.2 |
| 1.203 | 5.4 |
| 1.286 | 5.5 |
| 1.687 | 5.7 |

## 2024-T3 Bare Alumimum

| Plate <br> Hrumber | Thickness <br> $($ in. $)$ | Tensile Load <br> PSI | Transverse Load <br> LB. |
| :---: | :---: | :---: | :---: |
| 1a |  |  |  |
| 2a | 0.050 | 10,000 | 60 |
| $3 a$ | 0.050 | 10,000 | 80 |
| $4 a$ | 0.050 | $n$ | 80 |
| $5 a$ | 0.050 | $n$ | 100 |
| $6 a$ | 0.050 | $n$ | 100 |
| $7 a$ | 0.050 | $n$ | 125 |
| $8 a$ | 0.050 | $n$ | 125 |
| $9 a$ | 0.050 | $n$ | 150 |
| $10 a$ | 0.050 | $n$ | 150 |
| $11 a$ | 0.050 | $n$ | 175 |
| $12 a$ | 0.050 | $n$ | 175 |
| $13 a$ | 0.050 | $n$ | 200 |
|  | 0.050 |  | 200 |

Plate tla

| 2 a |  |
| :---: | :---: |
| 0.306 | 0 |
| 0.331 | 12 |
| 0.361 | 30.7 |
| 0.404 |  |
| 0.455 |  |
| 0.491 | 81. |
| 0.573 | 100 |
| 0.615 | 112. |
| 0.672 | 130.6 |
| 0.749 | 145.7 |
| 0.798 | 160.0 |
| 0.883 | 179.8 |
| 0.93 |  |


| Plate ${ }^{2}$ |  |
| :---: | :---: |
| 2a | $\mathrm{NrIO}^{-}$ |
| 0.324 | 0 |
| 0.348 | 0.0 |
| 0.393 | 20.0 |
| 0.432 | 28.9 |
| 0.491 | 37.2 |
| 0.546 | 44.1 |
| 0.613 | 51.4 |
| 0.677 | 7.0 |
| 0.744 | 62. |
| 0.826 | 67.6 |
| 0.897 | 72.4 |
| 0.954 | 75.4 |
| 1.031 | 79.5 |
| 1.084 | 82.3 |
| . 161 | 85.9 |
| 1.221 | 88.3 |
| 1.287 | 91.6 |
| 1.369 | 94.3 |
| 1.434 | 96.8 |
| 1.524 | 100.0 |
| 1.595 | 102.9 |
| 1.717 | 106.9 |
| 1.794 | 110.0 |
| 1.856 | 113. |

Plate 43

| 2. | 昷10 |
| :---: | :---: |
| 0.316 | 0.0 |
| 0.333 | 10.0 |
| 0.368 | 17.0 |
| 0.405 | 24. |

Plate 3 a cont.

| $\frac{2 a}{0.452}$ | $\frac{\text { 7610 }}{}{ }^{3}$ |
| :--- | ---: |
| 0.511 | 36.0 |
| 0.560 | 43.6 |
| 0.616 | 49.0 |
| 0.688 | 54.3 |
| 0.754 | 60.0 |
| 0.826 | 64.9 |
| 0.883 | 68.7 |
| 0.953 | 72.4 |
| 1.012 | 76.5 |
| 1.121 | 81.4 |
| 1.158 | 85.8 |
| 1.235 | 90.2 |
| 1.280 | 94.0 |
| 1.366 | 99.0 |
| 1.447 | 103.0 |
| 1.537 | 106.9 |
| 1.612 | 110.0 |
| 1.704 | 114.5 |
| 1.814 | 119.0 |

Plate 44e

| $\frac{2 a}{0}$ |  |
| :--- | ---: |
| 0.376 |  |
| 0.10 .0 |  |
| 0.452 | 7.6 |
| 0.535 | 13.4 |
| 0.615 | 18.6 |
| 0.702 | 24.6 |
| 0.829 | 30.1 |
| 0.907 | 33.6 |
| 0.984 | 37.5 |
| 1.071 | 41.0 |
| 1.167 | 45.2 |
| 1.247 | 49.5 |
| 1.336 | $5 . .6$ |
| 1.419 | 58.6 |
| 1.497 | 61.8 |
| 1.615 | 67.1 |
| 1.685 | 71.1 |
| 1.785 | 74.0 |

Prate 45a

| 20 | Mr10 ${ }^{3}$ |
| :---: | :---: |
| 0.319 | 0.0 |
| 0.360 | 8.2 |
| 0.422 | 14.5 |
| 0.474 | 20.1 |

Plate \#5a cont.

| 2 a | $\underline{\mathrm{Nx} 10^{3}}$ | 2 a | $\underline{\mathrm{NX}} 1 \mathrm{O}^{3}$ |
| :---: | :---: | :---: | :---: |
| 0.545 | 26.5 | $\overline{0.306}$ | 0.0 |
| 0.619 | 31.1 | 0.332 | 2.3 |
| 0.714 | 36.0 | 0.358 | 5.0 |
| 0.773 | 39.2 | 0.400 | 8.4 |
| 0.846 | 43.8 | 0.429 | 11.5 |
| 0.934 | 49.1 | 0.496 | 15.1 |
| 1.001 | 52.5 | 0.648 | 18.5 |
| 1.077 | 55.4 | 0.604 | 21.4 |
| 1.161 | 58.6 | 0.664 | 24.6 |
| 1.216 | 60.7 | 0.748 | 28.7 |
| 1.299 | 64.6 | 0.812 | 31.4 |
| 1.385 | 68.5 | 0.891 | 34.4 |
| 1.479 | 71.8 | 0.969 | 36.8 |
| 1.555 | 73.8 | 1.036 | 39.2 |
| 1.612 | 75.5 | 1.085 | 41.6 |
| 1.722 | 79.0 | 1.148 | 44.4 |
| 1.835 | 82.0 | 1.232 | 47.4 |
| 1.922 | 84.8 | 1.324 | 51.3 |
|  |  | 1.391 | 56.3 |

Plate \# 6 a

| 2a | $\mathrm{Nx} 10^{3}$ | Plate 48a |  |
| :---: | :---: | :---: | :---: |
| 0.306 | 0.0 | 20 |  |
| 0.334 | 2.6 | 0.298 | 0.0 |
| 0.347 | 5.1 | 0.320 | 1.9 |
| 0.403 | 9.8 | 0.359 | 3.9 |
| 0.443 | 13.3 | 0.388 | 6.4 |
| 0.491 | 16.7 | 0.439 | 9.4 |
| 0.537 | 19.3 | 0.508 | 12.3 |
| 0.597 | 22.7 | 0.566 | 15.2 |
| 0.635 | 25.2 | 0.625 | 17.8 |
| 0.691 | 27.5 | 0.693 | 20.8 |
| 0.743 | 29.4 | 0.751 | 23.7 |
| 0.821 | 32.8 | 0.815 | 25.9 |
| 0.884 | 35.7 | 0.899 | 29.2 |
| 0.350 | 38.0 | 0.971 | 32.3 |
| 1.013 | 40.3 | 1.027 | 36.2 |
| 1.089 | 43.0 | 1.080 | 40.0 |
| 1.126 | 44.8 |  |  |
| 1.223 | 47.8 | Prate f9a |  |
| 1.309 | 50.5 |  |  |
| 1.354 | 52.6 |  |  |
| 1.459 | 55.0 | 28 | H20 |
| 1.558 | 57.3 | 0.288 | 0.0 |
| 1.661 | 59.3 | 0.311 | 1.8 |
| 1.739 | 61.8 | 0.332 | 3.4 |
|  |  | 0.370 | 5.9 |
|  |  | 0.404 | 8.8 |
|  |  | 0.453 | 10.9 |
|  |  | 0.507 | 13.2 |


| 2. | $\underline{N T 10^{3}}$ | 2 a | N-10 ${ }^{3}$ |
| :---: | :---: | :---: | :---: |
| 0.567 | 16.4 | 1.332 | 41.6 |
| 0.598 | 18.5 | 1.461 | 43.5 |
| 0.682 | 21.3 |  |  |
| 0.725 | 22.6 |  |  |
| 0.774 | 24.3 | plate | 12 |
| 0.844 | 27.1 |  |  |
| 0.915 | 29.6 | 2 m | $\underline{\mathrm{Na}} 10^{3}$ |
| 0.969 | 31.9 | 0.310 | 0.0 |
| 2.041 | 34.5 | 0.334 | 2.5 |
| 1.120 | 36.5 | 0.385 | 4.9 |
| 1.176 | 38.4 | 0.437 | 8.2 |
| 1.248 | 42.7 | 0.504 | 10.9 |
|  |  | 0.576 | 13.9 |
|  |  | 0.627 | 17.2 |
| Plate | H10a | 0.724 | 21.2 |
|  |  | 0.890 | 24.1 |
| 2a | $\underline{1 \times 10}$ | 0.987 | 26.2 |
| 0.294 | 0.0 | 1.057 | 28.8 |
| 0.312 | 1.3 | 1.233 | 30.2 |
| 0.360 | 3.7 | 1.592 | 31.0 |
| 0.406 | 6.3 |  |  |
| 0.458 | 8.6 | Prate |  |
| 0.525 | 10.9 |  |  |
| 0.585 | 13.2 |  |  |
| 0.629 | 14.6 | 2 a | $\underline{M \times 10^{3}}$ |
| 0.665 | 17.6 | 0.304 | 0.0 |
| 0.784 | 19.8 | 0.330 | 2.3 |
| 0.842 | 22.7 | 0.380 | 4.1 |
| 0.979 | 29.3 | 0.428 | 6.3 |
|  |  | 0.474 | 8.2 |
|  |  | 0.550 | 11.7 |
| Plate flla |  | 0.600 | 14.7 |
| $\frac{20}{0.283}$ | $\underline{\mathrm{Rax}} 0^{3}$ | 0.688 | 17.5 |
|  |  | 0.751 | 20.0 |
| 0.283 | 0.0 | 0.803 | 22.9 |
| 0.304 | 1.6 | 0.862 | 26.1 |
| 0.325 | 3.1 | 0.940 | 29.2 |
| 0.364 | 5.8 | 1.008 | 32.7 |
| 0.406 | 8.5 | 1.070 | 37.8 |
| 0.450 | 10.2 |  |  |
| 0.489 | 12.1 |  |  |
| 0.538 | 15.3 |  |  |
| 0.574 | 17.4 |  |  |
| 0.641 | 20.9 |  |  |
| 0.737 | 23.4 |  |  |
| 0.797 | 27.4 |  |  |
| 0.889 | 30.3 |  |  |
| 0.942 | 33.3 |  |  |
| 1.011 | 34.7 |  |  |
| 1.069 | 35.9 |  |  |
| 1.160 | 38.8 |  |  |
| 1.233 | 40.6 |  |  |

```
\(2024-13\)
```

| Thicknes a (in.) | Yield Strength (at $0.2 \%$ - ESI) | Tensile Strength (ESI) | 务 Niongation ( $2^{\prime \prime}$ gage length) |
| :---: | :---: | :---: | :---: |
| 0.050 clad | 39.7 | 61.7 | 12.5 |
| 0.050 Bare | 56.4 | 75.5 | 17.4 |
| 0.080 Bare | 49.9 | 77.3 | 18.7 |
| 0.100 Bare | 61.8 | 80.3 | 18.2 |
| 0.100 Clad |  | Hot Available |  |
| 0.125 Bare | 51.4 | 78.2 | 19.5 |
| 0.125 clad |  | Hot Available |  |
| 0.160 clad | 48.6 | 69.2 | 11.6 |
| 7075-96 |  |  |  |
| $\begin{aligned} & \text { Thickness } \\ & \text { (in.) } \end{aligned}$ | Yield Strength (at 0.02\% - KSI) | Tensile Strength (ISI) | \% Elongation ( $2^{\prime \prime}$ gage length) |
| 0.050 clad | 63.1 | 77.5 | 7.8 |
| 0.100 Clad | 65.6 | 71.3 | 2.2 |
| 0.120 Bare | 75.4 | 84.2 | 9.5 |

