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SATURN V/SKYLAB MULTIPLE-PROTUBERANCE HEAT TRANSFER TEST DATA ANALYSIS

By W. L. Shaffer, C. P. Williamson, and C. R. Mullen

The Boeing Company Huntsville, Alabama

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16. AØSTRACT		·		<u></u>
Multiple protuberance h	eat transfer data	obtained from	n wind tunnel t	ests con-
ducted at Langley Research C	lenter are present	ed. The prim	ary purpose of	the test
series was to determine the	heat transfer to	bodies submer	ged in the wake	e of, or
having their surface impinge	d upon by the sho	ock or shocks	of one or more	protuber-
ances. Data were taken for	Mach numbers of 2	.49. 3.51 and	4.44 at a Revn	olds
number of $3 \times 10^6/\text{ft}$ Protu	berance models co	insisted of (1) four models w	rith a
15 or 30 degree forebody co	netant cross-sect	ion center bo	$\frac{1}{2}$ and a $15-3$	lon or
$\begin{array}{c} 15 \text{ or } 50 \text{ degree forebody}, co \\ 00 \text{ degree afterbody} (2) = 1 \end{array}$	A-dogree romp or	d(3) four mode	dolg of anti-fl	nttor
bet continue. The models re-	4-degree ramp, an	turnol addat	acts of anci-in	in cincu
hat sections. The models we	re mounted on the	tunnel sidew	all test plate	in singu-
lar and multiple configurati	ons and heat tran	ister data rec	orded from ther	mocouples
located on the test plate an	d models.			
				.1 . 1 .
Additionally, test plat	e pressures were	obtained in the	ne vicinity of	the hat
section models. Some limite	d comparisons of	theoretical h	eat transfer ar	d the
test data have been made.				
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NOMENCLATURE

d	Protuberance height
Н	Step height
h	Heat-transfer coefficient
₽ ħ _C	Average heat-transfer coefficient in gap
h _{cL}	Heat-transfer coefficient at $X = L$
h _O	Heat-transfer coefficient, clean body
h _o '	Average clean-body heat-transfer coefficient at protuberance location
K	Thermal conductivity
L	Running length
M	Mach number
Pp	Plateau pressure
R _e	Reynolds number
Х	Distance from plate leading edge
Y	Distance from plate centerline
Z	Vertical distance
δ	Boundary 1ayer thickness

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1.0 INTRODUCTION

This doucment contains an analysis of heat-transfer and pressure data (Reference 1) obtained during a multiple-protuberance heating test (Project 925) conducted at the Langley Research Center, July and August of 1970. The tests were conducted at Mach numbers of 2.49, 3.51 and 4.44 in the Unitary Plan Wind Tunnel with a nominal Reynolds number per foot of $3x10^6$.

The primary purpose of the test program was to investigate the effect of aerodynamic heating in the shock impingement and wake heating regions on and around multiple-protuberance configurations submerged in a turbulent boundary layer. The protuberance models were arranged to allow measurement of induced heating rates associated with single and multiple-protuberance disturbances.

Existing McDonnell-Douglas protuberance models (References 1 through 3) were refurbished and mounted to a new instrumented test plate in various configurations.

The data have been plotted to show the heating effects of multiple-protuberances and have been compared with analytical calculations and other data.

2.0 GENERAL TEST CONDITIONS AND ANALYSIS

The multiple-protuberance test incorporates the use of nine separate models (five instrumented) mounted in different combinations to an instrumented flat plate to form 19 different configurations. These configurations are shown in Figure 2-1 (configuration 1, the clean plate, is not shown).

All protuberance models and the test plate are shown schematically with dimensions in Figures 2-2, 2-3 and 2-4. The general protuberance shapes include a 15 or 30 degree forebody, constant cross-section centerbody, and a 15, 30 or 90 degree afterbody. Wind tunnel installation photographs of some configurations are shown in Figures 2-5 through 2-9. Additional details on the fabrication and instrumentation of the models and test plate are given in Reference 1.

The multiple protuberance configurations were tested at Mach numbers of 2.49, 3.51 and 4.44 at a nominal Reynolds number per foot of 3.0x10⁶. The total temperature and pressure ranges were 610°R to 713°R and 2799 psf to 8686 psf respectively. The test schedule is presented in Table 2-I and contains tunnel total pressure, total temperature, Mach number, unit Reynolds number, and a description of the configuration for each test run. Pressure measurements were obtained for configuration 11, and heat-transfer measurements for all other configurations.

The models were mounted on the test plate which was a part of the tunnel wall during tests. The models were therefore immersed in the turbulent boundary layer of the tunnel wall. The boundary layer profiles were not measured during this test but have been measured in the past and are contained in Reference 2. These boundary layer profiles have been used in the analytical calculation (profiles do not agree with 1/7 power law distribution).



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONDIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Continued)



FIGURE 2-1. MULTIPLE-PROTUBERANCE TEST CONFIGURATIONS (Concluded)









MODEL 5



FIGURE 2-2. SCHEMATIC DRAWINGS OF GENERAL PROTUBERANCE MODELS





FIGURE 2-3. SCHEMATIC DRAWINGS OF HAT SECTION PROTUBERANCE MODELS



FIGURE 2-4. TEST PLATE, MODEL ATTACH POINT LOCATIONS AND NOMENCLATURE









FIGURE 2-8. PHOTOCRAPH OF CONFIGURATION 8 INSTALLATION





(Continued)
2-I.
TABLE

LANGLEY MULTIPLE-PROTUBERANCE TEST RUN SCHEDULE

PARAMETER INVESTIGATED	M ₁₃ induced pressure field (pressure measurements during Rum 1 only)	M13 and M_{14} induced heating	Configuration 12 repeated due to model difficulties during Rum 2.	MIS and MI6 induced heating	M4 reference values	M4 shock impingement on M1 forebody	M ₁ reference values	M4 shock impingement on M1 afterbody
TOTAL PRESS. PSF	2799 2550 4241 7267	3237 5406 8673	3199 5318 8670	3211 5413 8651	5377 8511 3171	3187 5376 8690	3200 5385 8680	3176 5330 8654
TOTAL TEMP.	610 635	708 706 678	709 689 684	709 707 689	697 676 696	708 702 687	712 706 687	700 700 690
REYNOLDS-NO./ FTx106	3.14 3.14 3.00 3.00	3.12 3.00 3.25	3.07 3.06 3.20	3.10 3.00 3.16	3.05 3.19 3.13	3.08 3.01 3.07	3.06 2.99 3.18	3.11 3.01 3.14
MACH No.	2.49 2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	3.51 4.44 2.49	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44
CONFIGURATION	11	12	12	13	14	15	19	16
RUN POINT	1. 5408	2-1	3-2	4	20 21 21	6-1 32	7-6 7 8	8-1 3 3

M1, M2, M4, M5 and M12 are instrumented except for M1 which is only instrumented on runs 6, 7 and 8. Stringer number 11 instrumented with thermocouples on runs 2 and 3. Stringer number 9 instrumented with pressure orifices on run 1. NOTES:

TABLE 2-1. (Continued)

SCHEDULE
RUN
TEST
MJLTIPLE-PROTUBERANCE
LANGLEY

PARAMETER INVESTIGATED	M ₄ reference values	Clean plate reference values	Effect of stringers on Configuration 8	M5 in M1/M2 combination wake	M5 in M1 wake	M5 reference values	M2 reference values	M1 wake
TOTAL PRESS. PSF	3189 5391 8686	3179 5373 8668	3220 5401 8658	3219 5390 8591	3207 5402 8612	3209 5392 8630	3219 5351 8561	3207 5386 8606
TOTAL TEMP.	989 206 669	713 710 684	710 711 689	714 713 × 690	715 711 683	718 712 689	718 711 685	715 711 693
REYNOLDS NO./ FTX106	3.13 2.99 3.18	3.04 2.96 3.20	3.09 2.98 3.17	3.07 2.95 3.11	3.05 2.98 3.19	3.04 2.97 3.15	3.05 2.95 3.14	3.05 2.97 3.11
MACH NO.	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44
CONFIGURATION	17	1	σι	œ	w	Q	6	4
RUN POINT	9-1 375	10-1	11-1 3 3	12-1 3,2,1	13-1 3 3	14-1 3 3	15-1 3 3	16-1 2 3

M1, M2, M4, M5 and M12 are instrumented except for M1 which is only instrumented on runs 6, 7 and 8. Stringer number 11 instrumented with thermocouples on runs 2 and 3. Stringer number 9 instrumented with pressure orifices on run 1. NOTES:

PARAMETER INVESTIGATED	Mg in M1 wake	M12 in M1 wake	M12 reference values	Plate with stringers reference values	
TOTAL PRESS. PSF	5207 5380 8611	3214 5388 8626	3216 5387 8616	3222 5380 8666	
TOTAL TEMP.	711 707 689	714 709 680	717 713 686	712 712 691	
REYNOLDS NO./ FTx106	3.08 2.99 3.15	3.06 2.97 3.23	3.05 2.95 3.16	3.08 2.95 3.14	
MACH NO.	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	2.49 3.51 4.44	
CONFIGURATION	18	£	~	10	
POINT	10.10	18-1	19-1	20-1 3 3	

(Concluded)	
2. I.	
TABLE	

LANGLEY MULTIPLE-PROTUBERANCE TEST RUN SCHEDULE

 M_1 , M_2 , M_4 , M_5 and M_12 are instrumented except for M_1 which is only instrumented on runs 6, 7 and 8. Stringer number 11 instrumented with thermocouples on runs 2 and 3. Stringer number 9 instrumented with pressure orifices on run 1. NOTES:

3.0 TEST PLATE ENVIRONMENT

Heating environments on the clean plate and stringer plate configurations and the protuberance effects on the test plate are discussed in the following paragraphs. A complete tabulation of test data is presented in Reference 1. Heat-transfer coefficient ratios for the test plate are presented in Appendix A for all test configurations and Mach numbers.

3.1 CLEAN PLATE AND PLATE WITH STRINGERS

Heat-transfer coefficients on the clean plate centerline are presented in Figure 3-1 as a function of distance from the plate leading edge. Corresponding measurements from a previous protuberance test (Reference 3) are included for comparison. The differences between present measurements and those presented in Reference 3 are attributable to data accuracy limits and variations in test surface flatness.

Theoretical flat plate heat-transfer coefficients were determined by the method of Van Driest for turbulent flow (References 4 and 5). The theoretical values are based on tunnel conditions, measured wall temperatures, a 5.0 inch boundary layer thickness, and Reynolds number per foot of 3.0x10⁶. The Langley Unitary Plan Wind Tunnel test section boundary layer profiles used in the analysis were taken from Reference 2. Analytically determined heat-transfer coefficients are compared with the test results in Figure 3-2. The calculated coefficients exceed the test results at all Mach numbers. Heat loss through the support structure, which reduces the experimentally determined heat-transfer coefficient, was estimated to be about 10 percent. However, this accounts for only a portion of the difference between experimental data and analysis. Previous investigators (Reference 6) have reported similar non-correlation of analytical calculations and Langley protuberance test data. Reference 6 indicates that clean plate data correlation is improved by using a calculated recovery temperature (T_r) rather than the measured equilibrium temperature (T_e) in the data reduction equations. This procedure effectuates increases in the experimentally determined heat-transfer coefficients by a factor of approximately $(T_e - T_w)/(T_r - T_w)$. Application of this correction factor to the present clean plate data resulted in increases in the measured heat-transfer coefficient ranging from about 10 percent at Mach number 2.49 to about 25 percent at Mach number 4.44 as shown in Figure 3-2.

Heat-transfer coefficients obtained on the plate with longitudinal stringers (configuration 10) are presented in Figure 3-3 compared to clean plate data for each Mach number. The stringers reduce the clean plate coefficients about 10 percent at $M_{\infty} = 2.49$, 25 percent at $M_{\infty} = 3.51$, and 35 percent at $M_{\infty} = 4.44$. This reduced heating effect may be partially due to disturbances caused by the stringer leading edges.

3.2 PLATE ENVIRONMENT WITH ATTACHED SINGLE AND MULTIPLE-PROTUBERANCE MODELS

Heat-transfer coefficients measured on the test plate with protuberance models attached were reduced to ratio form (h/h_0) by division of the measured value by the clean plate heat-transfer coefficient obtained at the same measurement location. These ratios are shown in Figures A-1 through A-48 of the Appendix. Presentation of the data in ratio form reduced the effects of test plate fabrication imperfections and uncertainties in the thermal properties of the test plate material.

The test plate surface heat-transfer coefficient ratios presented in Appendix A indicate significant increases in the regions influenced by protuberance forebody compressions and protuberance wakes.

Two dimensional free stream oblique shock angles have been superimposed on selected figures in the Appendix for comparison with the heat-transfer coefficient ratios. It is evident after studying these figures, that the increases in heating begin a little ahead of the free stream shock location and extend down stream. The thick tunnel wall boundary layer causes the shock to affect the surface forward of the free-stream shock location.

The test plate heat-transfer coefficient ratios (h/h_0) as a function of distance (X) along the test plate at a constant distance (Y) from the plate centerline are presented in Figures 3-4 through 3-12 for various single and multiple-protuberance combinations at each test Mach number. These distributions are considered to be representative of the prominant heating effects measured on the test plate surface.

Figures 3-4 through 3-6 compare the heat-transfer coefficient ratios obtained on the plate centerline (Y = 0) for a double-wedge model (Model 12) mounted in the center of Model 1 wake to those obtained with Model 1 wake alone and Model 12 wake alone. The maximum h/ho value for the Model 1 and Model 12 combination was greater at each Mach number than that for the Model 1 wake alone, and did not exceed a magnitude of 1.83. These increases occurred near the Model 12 base and did not persist over appreciable distances along the plate. At extreme downstream measurement locations the influence of Model 12 on the Model 1 wake was small. The double-wedge model generated less of a heating effect than Model 1 due to the double-wedge height of 0.6325 inches compared to Model 1 height of 5.0 inches. The h/ho values in the Model 1 (15 degree afterbody) wake did not consistently increase with free-stream Mach number and exhibited a maximum value of 1.66 at M_{∞} = 2.49. The heat-transfer coefficient ratio distributions obtained at the higher Mach numbers are, in general, more irregular (compare Figures 3-4 through 3-6) because of the decreased data accuracy at the higher Mach numbers.

3.2 (Continued)

Other configurations tested which consisted of a single protuberance model mounted in the proximity of the Model 1 wake did not generally increase the h/h_0 magnitudes above those obtained in the Model 1 single wake.

Figures 3-7 through 3-9 compare the plate heat-transfer coefficient ratios obtained at locations 8.0 inches from the plate centerline (Y = 8.0) for Model 5 mounted in the aft double wake of Models 1 and 2 to those for Model 2 wake alone and Model 5 alone. Model 5 mounted in the double wake of Models 1 and 2 produced h/h_0 magnitudes which exceeded those obtained at the same measurement locations in the Model 2 single wake or in the single Model 5 forebody compression region. However, this increase was not present at all locations upstream of Model 5. The maximum h/h_0 values obtained in the double wake region near Model 5 were about equal to those obtained at locations further upstream in the Model 2 single wake (approximately 2.0).

The single wake of Model 2 (30 degree afterbody) induced maximum h/h_0 values which did not consistently increase with free-stream Mach number, and did not exceed a magnitude of 2.08.

Figures 3-10 through 3-12 compare the plate heat-transfer coefficient ratios obtained for Model 4 mounted in the forebody compression region of Model 1 to those for Model 4 alone. The largest heating influence occurred near the forebody-centerbody junction of Model 4, and increased consistently with increasing free-stream Mach number to $h/h_0 = 2.20$ at $M_{\infty} = 4.44$. The maximum heating rate obtained in the Model 4 (30 degree forebody) compression region was about equal for Model 4 single and Model 4 and Model 1 mounted in combination.

Heating environments were also obtained with Model 4 mounted in the afterbody expansion region of Model 1 (configuration 16). As seen in Appendix A, Figures A-37 through A-42, the heating effect of Model 4 forebody compression was decreased from that for Model 4 alone by the flow expansion around the Model 1 centerbody-afterbody junction.

The single Model 1 (15 degree forebody) compression effects are given in Appendix A, Figures A-46 through A-48 (configuration 19). The maximum h/h_0 magnitude induced by the 15 degree forebody also occurred near the forebody-centerbody junction, but was lower than that for the 30 degree forebody of Model 4. The influence of Model 4 did not significantly increase the h/h_0 magnitudes in the proximity of Model 1 forebody.

Theoretical values for the test plate heat-transfer coefficient ratios induced by the 15 and 30 degree forebody shocks were estimated using Van Driest's turbulent flat plate theory and flow properties behind the forebody shock. These estimates were divided by the theoretical clean plate values given in Figure 3-2. The calculated ratios are shown in Figure 3-13 as a function of free-stream Mach number for 15 and 30 degree forebodies. Included in Figure 3-13 are the single model ratios from measurements for Model 1, Model 5 and Model 4 (15, 15 and 30 degree forebodies, respectively). The experimental
3.2 (Continued)

values chosen for comparison are the peak ratios occurring along measurement locations 4.0 inches from the model centerline in the forebody compression region. The measured maximum ratios appear consistently lower than calculated values by 10 to 14 percent for the 15 degree forebody and 12 to 26 percent for the 30 degree forebody throughout the Mach number range. The calculation method thus yields valid effects of Mach number and forebody angle and heattransfer coefficient ratio magnitudes which compare favorably but are consistently greater than measured maximum ratios. No analyses were made to evaluate protuberance wake flow fields and consequent heating effects.

The effect of stringers on plate heating distributions with attached protuberances was generally to lower the ratios induced on the clean plate (Figures A-22 through A-27). At measurement locations where the stringer plate heat-transfer coefficient ratio was larger than the corresponding clean-plate ratio, the conversion to heat-transfer coefficient magnitude yielded in most cases a lower heating rate in the presence of stringers.



Po, BTU/FT²SEC °F



CLEAN PLATE HEAT-TRANSFER COEFFICIENT, ho, BTU/FT²SEC-°R

FREE STREAM MACH NUMBER, M_{∞}

FIGURE 3-2. CLEAN PLATE HEAT-TRANSFER COEFFICIENT VARIATION WITH MACH NUMBER - CONFIGURATION 1



EFFECT OF STRINGERS ON PLATE HEAT-TRANSFER COEFFICIENT VARIATION WITH DISTANCE ALONG PLATE - CONFIGURATION 10 FIGURE 3-3.



FIGURE 3-4. EFFECT OF DOUBLE WEDGE (MODEL 12) IN MODEL 1 WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATIONS 2, 3 AND 4



FIGURE 3-5. EFFECT OF DOUBLE WEDGE (MODEL 12) IN MODEL 1 WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATIONS 2, 3 AND 4



FIGURE 3-6. EFFECT OF DOUBLE WEDGE (MODEL 12) IN MODEL 1 WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATIONS 2, 3 AND 4

HEAT-TRANSFER COEFFICIENT RATIO, h/ho



FIGURE 3-7. EFFECT OF MODEL 5 IN MODEL 1 AND 2 DOUBLE WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATIONS 6, 7 AND 8



FIGURE 3-8. EFFECT OF MODEL 5 IN MODEL 1 AND 2 DOUBLE WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATIONS 6, 7 AND 8



FIGURE 3-9. EFFECT OF MODEL 5 IN MODEL 1 AND 2 DOUBLE WAKE ON PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATIONS 6, 7 AND 8



FIGURE 3-10. EFFECT OF MODEL 1 FOREBODY SHOCK COMPRESSION ON MODEL 4 INDUCED PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATIONS 14 AND 15



FIGURE 3-11. EFFECT OF MODEL 1 FOREBODY SHOCK COMPRESSION ON MODEL 4 INDUCED PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATIONS 14 AND 15



FIGURE 3-12. EFFECT OF MODEL 1 FOREBODY SHOCK COMPRESSION ON MODEL 4 INDUCED PLATE HEAT-TRANSFER COEFFICIENT RATIO DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATIONS 14 AND 15



FIGURE 3-13. HEAT-TRANSFER COEFFICIENT RATIO INDUCED BY SINGLE PROTUBERANCE FOREBODY SHOCK COMPRESSION AS A FUNCTION OF MACH NUMBER

4.0 PROTUBERANCE MODEL ENVIRONMENT

Heating environments on the protuberance models tested are presented and discussed in the following paragraphs. These environments were measured on both single-protuberance and multiple-protuberance configurations. A complete tabulation of test data is presented in Reference 1. The heat-transfer coefficients on protuberance models were divided by an average of the clean plate values (h_0') obtained in and near the test plate surface area covered by the model base in order to obtain heat-transfer coefficient ratios. These ratios are presented in Appendix A of this report. Test configurations are shown in Figure 2-1.

Heat-transfer coefficient data measured along the centerline of Model 5 are presented in Figures 4-1, 4-2 and 4-3 for test configurations 6, 5 and 8, respectively. Test data for Model 5 alone (configuration 6) are presented in Figure 4-1 alone with analytically determined heat-transfer coefficients. It is evident in Figure 4-1 that the analytical calculations and test data correlate in both magnitude and trends. The increase in heating with distance along the conical forebody is caused by the variation in boundary layer properties. If the protuberance were in a free stream flow the heating would decrease with distance along forebody. Analytical values were determined by the method of Van Driest (References 4 and 5) with local flow properties determined as follows:

a) Forebody Surface

An attached conical shock wave based on the tunnel upstream flow parameters is developed on the protuberance model forebody. Local flow properties in the boundary layer ahead of the shock wave are determined from boundary layer distributions presented in Reference 2 at the height on the forebody for which heating is to be evaluated. These properties are modified by passing them through the conical shock and these properties are used in the heating calculation.

b) Centerbody, Forward End

Flow parameters at the edge of the boundary layer behind the shock wave are expanded around the cone/cylinder junction with the use of the Prandtly-Meyer relationships. Local flow conditions are then determined.

c) Centerbody, Aft End

Flow at the aft end of the centerbody is assumed to have returned to free stream conditions.

4.0 (Continued)

A plot of heat-transfer coefficient ratios (protuberance heat-transfer coefficients divided by clean plate value) versus Mach number is presented in Figure 4-4 for points on the forebody and top of Model 5. Data are presented for test configurations 5, 6 and 8. A comparison of data from the three configurations indicates a decrease in heating to the Model 5 forebody when the protuberance is placed in the wakes of Models 1 and 2.

Model 4 centerline heat-transfer coefficient values are presented in Figures 4-5 and 4-6 for configurations 14 and 15, respectively. In these configurations, Model 4 is located at attach point 4. A minor increase in heating on Model 4 centerline is noted from the interference effects of Model 1 (a cross plot of Figures 4-5 and 4-6 are shown in Figure 4-7).

Heat-transfer coefficient ratios are presented in Figure 4-8 for points along the side of Model 4. Data are presented in Figure 4-8 for configurations 14 and 15. The effect of the Model 1 wake region on Model 4 side areas adjacent to Model 1 is to increase the heating to Model 4 slightly.

Heat-transfer coefficient ratios are presented in Figure 4-9 for points along the side of Model 1. The effects of shock impingement from the Model 4 forebody shock wave on the side of Model 1 centerbody are evident.

In test configurations 16, 17 and 18, Model 4 was located at attach point 5 (17 inches aft of attach point 4). Heat-transfer coefficients along Model 4 centerline are presented in Figure 4-10 for configuration 17 (Model 4 alone) and in Figures 4-11 and 4-12 for configurations 16 and 18, respectively (Model 4 in Model 1 wakes).

Heat-transfer coefficient ratios for Model 4 alone and Model 4 in Model 1 wake regions are presented in Figure 4-13 as a function of Mach number. Generally lower values are observed on Model 4 in Model 1 wake regions. The shock strengths from Model 1 are less than from Model 4 and would therefore have less effect on Model 4 than Model 4 does on Model 1.

Heat-transfer coefficients along the centerline of Model 12 (double wedge) are presented in Figures 4-14 and 4-15 for configurations 2 and 3, respectively. Comparison of Figures 4-14 and 4-15 indicates that the effect of the aft wake of Model 1 (configuration 3) on Model 12 is to increase the heattransfer coefficient values.

Two basic model forebody angles were tested. These were the 15-degree forebody (Models 1, 3 and 5) and the 30-degree forebody (Model 4). Data taken along the forebody centerline of Models 1, 3, 4 and 5 are plotted as a function of Z/δ in Figures 4-16, 4-17 and 4-18 for Mach numbers 2.49, 3.51 and 4.44, respectively. The test data indicate a reduction in forebody heating due to immersion in the boundary layer.



FIGURE 4-1. HEAT-TRANSFER COEFFICIENTS ON MODEL 5 CENTERLINE (CONFIGURATION 6)





NOTE: HEAT-TRANSFER COEFFICIENT MODEL 2 VALUES MEASURED ALONG MODEL 5 CENTERLINE ON MODEL 5 MODEL 1 FLOW-CONFIGURATION 8 FLOW - d = 5'' Х HEAT-TRANSFER COEFFICIENT - 10⁻³ BTU/FT²SEC-°R 10 MACH NO. O 2.49 3.51 △ 4.44 8 б 0 Ð \odot \odot 0 \odot 4 Ē Ĉ 2 2 Ŀ \triangle 0 5 6 1 4 2 3 0 X/d

FIGURE 4-3. HEAT-TRANSFER COEFFICIENTS ON MODEL 5 CENTERLINE (CONFIGURATION 8)



FIGURE 4-4. HEAT-TRANSFER COEFFICIENT RATIOS - MODEL 5 (CONFIGURATIONS 5, 6 AND 8)



FIGURE 4-5. HEAT-TRANSFER COEFFICIENTS ON MODEL 4 CENTERLINE (CONFIGURATION 14)





HEAT-TRANSFER COEFFICIENT - 10^{-3} BTU/FT²SEC-°F



FIGURE 4-7. HEAT-TRANSFER COEFFICIENT RATIOS - MODEL 4 (CONFIGURATIONS 14 AND 15)







HEAT-TRANSFER COEFFICIENT RATIO - h/ho'



FIGURE 4-10. HEAT-TRANSFER COEFFICIENTS ON MODEL 4 CENTERLINE (CONFIGURATION 17)



FIGURE 4-11. HEAT-TRANSFER COEFFICIENTS ON MODEL 4 CENTERLINE (CONFIGURATION 16)



FIGURE 4-12. HEAT-TRANSFER COEFFICIENTS ON MODEL 4 CENTERLINE (CONFIGURATION 18)



FIGURE 4-13. HEAT-TRANSFER COEFFICIENT RATIOS - MODEL 4 (CONFIGURATIONS 16, 17 AND 18)



HEAT-TRANSFER COEFFICIENT - 10-3 BTU/FT2SEC-°R



HEAT-TRANSFER COEFFICIENT - 10-3 BTU/FT2SEC-°R



FIGURE 4-16. HEAT-TRANSFER COEFFICIENTS ON MODEL FOREBODIES - MODELS 1, 2, 4 AND 5 (M = 2.49)



FIGURE 4-17. HEAT-TRANSFER COEFFICIENTS ON MODEL FOREBODIES - MODELS 1, 2, 4 AND 5 (M = 3.51)



FIGURE 4-18. HEAT-TRANSFER COEFFICIENTS ON MODEL FOREBODIES - MODELS 1, 2, 4 AND 5 (M = 4.44)

5.0 HAT SECTION AND GAP ENVIRONMENT

Hat section models (Models 13 through 16) were added as a part of the protuberance test program after analysis indicated that hat sections which had been placed between stringers on the S-IVB forward skirt would significantly increase the local heating environment. The main heating problem was not the local increases caused by protuberance effects but high heating on the stringers caused by gaps which existed between the antiflutter hat sections and the stringers. Because of the complexity of the flow field, this configuration was added to the protuberance test to verify the analysis.

Only one of the four hat section models (Model 13) tested, produced sufficient data for correlation purposes. The other hat section configurations provided only enough data to observe general trends (lack of instrumentation). These data are contained in Appendix A as heat-transfer coefficient ratios (Figures A-64 through A-69) and in Reference 1 as basic coefficients. The following paragraphs describe the heating and pressure data obtained on the plate around the hat section and in the gaps between stringers and hat sections.

The Model 13 hat sections were 0.5 inch in height and 0.5 inch wide (Figure 2-3) and are mounted between stringers as shown in Figure 2-1 (configurations 11 and 12). The hat sections are immersed in a 5-inch turbulent boundary layer. Figures 5-1, 5-2, and 5-3 show the pressure coefficient distributions on the plate caused by the hat sections (configuration 11) for Mach numbers of 2.49, 3.51 and 4.44. It is evident that there is a significant pressure increase ahead of the hat sections which diminishes with successive hat sections as would be expected with losses in total pressure with each hat section encountered. The pressure rise is well below that which would be caused by a normal shock (using Mach number at top of hat) and corresponds to that which would exist under flow separation conditions ahead of the hat sections. Figure 5-4 shows a comparison of the pressure data ahead of the first hat section compared with data from Reference 8. The pressure data of this test correlates with the other test data and shows a significant boundary layer effect. The pressure also seems to be more dependent on free-stream Mach number as the wall surface is approached. The equation (Reference 9) shown on Figure 5-4 along with the data is sufficient to provide a method of estimating pressures ahead of a forward facing step submerged in a boundary layer.

The flat plate heating distributions for the hat section, configuration 11, are shown in Figures 5-5, 5-6, and 5-7. There is a rapid rise in heating ahead of each hat section with the peak values ahead of each hat in the train being about the same. Behind each hat section the heating drops below flat plate values. The average value of heating over the entire plate with hat sections is approximately the same as if there were no hat sections (data oscillates around $h/h_0 = 1$). Data from various Y-coordinate locations have been plotted on these figures to more fully describe the interference region.

5.0 (Continued)

Pressure measurements were made on both the stringer and plate immediately ahead of the hat section and in the gap between the stringer and hat section. These data are plotted in Figure 5-8 in pressure coefficient form for all Mach numbers and for the first hat section in the train. It is evident at all Mach numbers that the pressure on the leeward side of the hat section is significantly lower than the front side pressures and indicates a choked flow condition (M = 1) in the gap. Figure 5-9 shows the heating on the stringer caused by the hat sections and gaps. The four double data points (at each Mach number) indicate the heating data within the gaps. The first two double data points are in the first hat section gap and the last two are in the second hat section gap. There is a significant heating gradient within the gap and the stringer heating ahead of and in the hat section gap is significantly above the clean plate heating.

Attempts have been made to correlate analytical calculations with the measured test data in the hat section gap. Since the flow in the gap was choked, it was assumed the boundary layer was laminar. The familiar Colburn equation for laminar flow was used with measured gap pressure to evaluate the heat-transfer coefficient. Figure 5-10 shows a comparison of the measured gap heat-transfer coefficient to the analytical calculation of an average gap coefficient and also the coefficient at X = L where L is the gap length. It is evident that the analytical calculations were determined by evaluating the total temperature in the boundary layer at the top of the hat section using the boundary layer data of Reference 2. The static temperature in the gap was then evaluated using the total temperature at the top of the hat and a Mach number of 1. The transport properties of the air in the gap were then evaluated using the static temperature in the gap.


FIGURE 5-1.

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BKESSURE COEFFICIENT

PLATE SURFACE PRESSURE COEFFICIENT DISTRIBUTION WITH STRINGERS AND HAT SECTIONS - CONFIGURATION 11 - M = 3.51 FIGURE 5-2.

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FIGURE 5-4. BOUNDARY LAYER EFFECTS ON PLATEAU PRESSURE AHEAD OF FIRST HAT SECTION (MODEL 13)



HEAT-TRANSFER COEFFICIENT RATIO - $h/\hbar_{\mathbf{O}}$

PLATE SURFACE HEAT-TRANSFER COEFFICIENT DISTRIBUTION WITH STRINGERS AND HAT SECTIONS - CONFIGURATION 11, M = 2.49FIGURE 5-5.



HEAT-TRANSFER COEFFICIENT RATIO - \hbar/\hbar_{O}

PLATE SURFACE HEAT-TRANSFER COEFFICIENT DISTRIBUTION WITH STRINGERS AND HAT SECTIONS - CONFIGURATION 11, M = 3.51 FIGURE 5-6.



HEAT-TRANSFER COEFFICIENT RATIO - h/h_{F.P}.

PLATE SURFACE HEAT-TRANSFER COEFFICIENT DISTRIBUTION WITH STRINGERS AND HAT SECTIONS - CONFIGURATION 11, M = 4.44 FIGURE 5-7.



PRESSURE COEFFICIENT



HEAT TRANSFER COEFFICIENT, BTU/FT2SEC "R



HEAT-TRANSFER COEFFICIENT - BTU/FT²SEC °R

5-12

6.0 CONCLUSIONS

Flat Plate

- 1. The magnitude of heat-transfer coefficients measured on the clean plate surface were in reasonable agreement with those available from previous investigations, but were less than those predicted analytically.
- 2. The heat-transfer coefficient magnitude obtained on the plate with longitudinal stringers was reduced by 10 percent to 35 percent compared to the clean plate.
- 3. Multiple-protuberance interference heating effects in general were no more severe than for a single protuberance although a few single measurements on some multiple configurations were observed to have higher heating conditions. The distribution and surface areas affected by the increased heating have been modified. The maximum heat-transfer coefficient ratios (interference coefficient divided by clean plate coefficient) in the interference regions were approximately 2.0.
- 4. The plate heat-transfer coefficient magnitude obtained with multiple protuberance models with longitudinal stringers was generally less than that obtained for the equivalent model arrangement without stringers.

Protuberances

- 1. Protuberance forebody heating is significantly affected by the depth of immersion in the boundary layer. Heating in the lower portions of the boundary layer is considerably less.
- 2. Protuberance forebody heating decreases when the forebody is immersed in the aft wakes of other protuberances.
- 3. Analytically determined heat-transfer coefficients on a single protuberance forebody and centerbody compare favorably with measured values. No attempt was made to analytically determine flow properties or heating in protuberance wake regions.
- 4. Protuberance created shock waves which impinge on an adjacent protuberance will increase heating to that protuberance. The larger the forebody angle the stronger the shock and the greater the increase in heating.

6.0 CONCLUSIONS (Continued)

Hat Section and Gap Heating

- 1. Significant increases in heating and pressure are realized on the surface ahead of hat sections (H/ δ = 0.1) placed perpendicular to the flow between stringers. The pressure rise has been correlated with empirical methods.
- 2. Pressures ahead of and behind the gap indicate a choked flow condition (M = 1) in the gap.
- 3. The heating in the gap between the stringers and hat sections can be significantly higher than clean plate heating. Laminar boundary layer theory with appropriate thermodynamic properties and length assumption and using measured pressure has been correlated with the heating in the gap. Peak heating in the gaps is reduced with successive downstream hat sections.

APPENDIX A

HEAT-TRANSFER COEFFICIENT RATIOS FOR TEST PLATE AND PROTUBERANCE MODELS





•1.34 •1.30 •1.26 •1.24 •1.24 •1.28 •1.18 •1.28 •1.26 •1.28 •1.29 •1. •1.38 •1.40 •1:40 01.41 Values presented are in units of BTUx10³/ft²sec-^oR ●1.40 ●1.45 ●1.45 1.38 •1.23 •1.25 ۰ •1.41 **@1.36** •1.28 ●1.35 •1.32 •1.26 •1.31 •1.29 ●1.41 • •1.44 •1.48 •1.44 •1.43 •1.42 •1.13 •1.32 •1.42 ●1.48 • 1.39 • 1.38 •1.37 •1.38 •1.39 1.44 ●1.28 ●1.27 ●1.49 •1.45 •1.55 .1.46 NOTE: 1.42 • •1.45 1.35 • •1.33 1.30 1.24 1.24 •1.28 •1.50 1.33 1.35 01.34 01.32 01.39 1.32 1.36 1.39 1.32 1.36 1.35 01.35 01.33 01.23 01.33 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 22.10 • •1.26 ●1.47 ●1.48 01.36 01.36 01.39 01.35 01.36 0 1.41 1.38 1.37 • • • • • • 1.37 •1.41 1.40 1.42 1.42 •1.28 •1.26 1.35 •• 1.41 1.31 • •1.32 •1.35 1.41 • • 1.43 1.28 •1.24 •1.36 1.27 •1.24 •1.35 1,30 1,31 1,27 1,27 12 SCALE: INCHES 1.34 1.36 1.39 • • • 1.36 1.34 1.38 19 Ö •1.27 •1.27 •1.49 1.32 1.32 •1.23 1.38 1.26 1.38 1.28 1.32 . 0 • .

CLEAN PLATE HEAT-TRANSFER COEFFICIENT DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 1 FIGURE A-2.







EFFECT OF DOUBLE WEDGE (MODEL 12) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 CONFIGURATION 2 FIGURE A-4.

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EFFECT OF DOUBLE WEDGE (MODEL 12) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 CONFIGURATION 2 FIGURE A-6.

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EFFECT OF DOUBLE WEDGE (MODEL 12) IN MODEL 1 WAKE ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 3 FIGURE A-8.





	•0.99	86°0. 90 Af	04 ●0.95 ●0.94 ●0.94 • ●0.94	94 0.95	32	33 ●1.0 3	58 @1.40 @1.35 @1.34 @1.37 @1.36 @1.35 @1.34 @1.3 1,3	•1.22	35	●0.94	●0.94	presented are the ratio of sat-transfer coefficient to m plate heat-transfer tent obtained at the same 1.
16.0●	0.94 0 .99	€0.93 0.90 ● €0.90 ●0.94 ●0.94	• •0.88 •0.92 •0.92 •0.93 •0.9	0.88 • • • • • • • • • • • • • • • • • • •	0.94 • •0.93 •0.91 •0.5	●1.04 ●1.04 ●1.04 ●1.04	el.48 el.45 el.43 el.43 el.	●1.20 ●1.20	5.0●			NOTE: Values p local he the clea coeffici location
	●0.88	0.87 0.85 0.85 0.83 0.83	0.88 0.194 0.79 0.80 0.83 0.81 0.79 0.80 0.83 0.81 0.79 0.80	0.76 0.79 0.75 0.84 0.87 0.889 0.89	0.70 0.70 .0.81	•0.82 92 92 92 92 92 92 92 92 92 92 92 93 92 93 93 93 93 93 94 94 94 95 95 95 95 95 95 95 95 95 95 95 95 95			● D. 84			-121
	86° 0 •			0.90 0.82 0.79 0.76 0.91 0.83 0.81 0.79	0.82 0.75 0.74 0.71 0.77 0.74 0.72	•0.57		0.78			•	SCALE: INCHES

FIGURE A-10. EFFECT OF MODEL 1 WAKE ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 4



FIGURE A-11. EFFECT OF MODEL 1 WAKE ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 4

A-12



FIGURE A-12. EFFECT OF MODEL 1 WAKE ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 4





A-14



















FIGURE A-18. EFFECT OF MODEL 5 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 6



FIGURE A-19. EFFECT OF MODEL 2 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 7



FIGURE A-20. EFFECT OF MODEL 2 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 7







EFFECT OF MODEL 5 IN MODELS 1 AND 2 AFT WAKES ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 8 FIGURE A-22.





A-24


FIGURE A-24. EFFECT OF MODEL 5 IN MODELS 1 AND 2 AFT WAKES ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 8





A-26



EFFECT OF MODEL 5 IN MODELS 1 AND 2 AFT WAKES ON STRINGER PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 9 FIGURE A-26.





F	•1.92	●I.86		•1.79	•1.89			•1.96						
•2.33	e2.13 e2.27 e2.15	•2.16	2.17 • • • 2.03 • 2.06 • 1.99 • 2.00	•2.12 •2.22 •2.19 •2.11 •2.08 •1.97 •2.00	2.13 •2.20 •2.04 •2.09 •2.08 •2.27 •2.06	2.00 • • 2.11 • 1.74 • 2.15	•2.15 •2.05 •2.08 •2.13	•2.25 •2.20 •2.16 •2.22 •2.15 •2.28 •2.01 •2.17 •2.12 •2.13	• 2.39 • 2.24 • 2.23 •		•2.62	•2.74	●2.64	NOTE: Values presented are in units of BTUx10 ³ /ft ² sec-°R
	•2.13			55 2.11 2.09 2.04 2.09 2.04	•2.10 •2.14 •2.05 •2.01 • • 2.13 •	•2.11 •1.44 •2.20 •1.94 •2.21	19 		2°2• 2°2• 2°2•	•2.30 •1.55	•2.75			
	•2.21		1.99 1.8 2.14 2		05 2.15 2.15 2.26 2.23 2.12 2.07 • 1.98 2.12 2.26 2.23 2.12 2.07 •	.32 2.08 2.11 2.14 1.75 2.14 2.14 2.14 2.14 2.14 2.14 2.14	•2.01 •2.16	• \$2.23 \$2.30 \$2.16 \$2.10	1.73 2.31 1.34 2.31 1.34 273224				٠	SCALE, INCRES

FIGURE A-28. STRINGER PLATE HEAT-TRANSFER COEFFICIENT DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 10





2.0•	• 0.70	0.57 00.54 00.62	•0.58		0.47 e 0.49 e 0.					of
0.67		• 0.57 • 0.54	e 0.47	P0.48	8 •0.46 •0.47 •0.47 • •0.43 •			•0.66	•0.63	are in units
•0.73 •0.68	•0.61	54 0.57 0.61 0.55 0.54 0.54 0.54	0.50 0.52 0.53 0.49	●0.48 ●0.46 ●0.47	•0.47 •0.49 •0.44 •0.47 •0.4 •0.44 •0.44		• 0.53			Values presented a BTUx10 ³ /ft ² sec-°R
•0.67	- 65 · 0	•0.58 •0.61 • • 0. 1 •0.56 •0.56 • •0.56 •	55 0.48 0.5080.24 0.56 0.55 0.55 31 0.51 0.46 0.50 0.57 49	64 05 20 15	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		• • 0* 55			NOTE:
	0.56	0.59 0.04 0.61	0.56 0.55 0.55 0.55 0.45 0.55 0.55 0.55 0.54 0.55	0.28 0.52 0.52	0.48 0.48 0.52 00 0.52 0.48 0.52 00 0.45 0.48 0.54 00 0.47 00 0.47 00 0.47 00 0.47 00 0.47 00 0.47 00 0.47 00 0.47 00 0.47 00 0.48 00 0.48 00 0.48 00 0.48 00 0.48 00 0.48 00 0.48 00 40 00 40 40 40 00 40 40 40 40 40 40 40 40 40 40 40 40 4	0 0 0 0 0 0 0				CHES 1 1 1 12
• • • • 72			0.52 0.55 0.55 0.50 0.50 0.50 0.50 0.50	0 0.48	●0.28 ● ●0.37					SCALE: IN C C C

FIGURE A-30. STRINGER PLATE HEAT-TRANSFER COEFFICIENT DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 10







FIGURE A-32. EFFECT OF MODEL 4 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 14







EFFECT OF MODEL 4 SHOCK IMPINGEMENT ON MODEL 1 FOREBODY ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 15 FIGURE A-34.







FIGURE A-36. EFFECT OF MODEL 4 SHOCK IMPINGEMENT ON MODEL 1 FOREBODY ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 15







EFFECT OF MODEL 4 SHOCK IMPINGEMENT ON MODEL 1 AFTERBODY ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 16 FIGURE A-38.







FIGURE A-40. EFFECT OF MODEL 4 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 17







FIGURE A-42. EFFECT OF MODEL 4 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 17







1 EFFECT OF MODEL 4 IN MODEL 1 WAKE ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 CONFIGURATION 18 FIGURE A-44.





A-465



FIGURE A-46. EFFECT OF MODEL 1 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 19







FIGURE A-48. EFFECT OF MODEL 1 ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 19





- NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 12 base (0.00264 Btu/ft²sec-°R).
- FIGURE A-49. EFFECT OF MODEL 1 WAKE ON DOUBLE WEDGE (MODEL 12) HEATING DISTRIBUTIONS AT MACH NUMBER 2.49





- NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 12 base (0.00133 Btu/ft²sec-°R).
- FIGURE A-50. EFFECT OF MODEL 1 WAKE ON DOUBLE WEDGE (MODEL 12) HEATING DISTRIBUTIONS AT MACH NUMBER 3.51





NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 12 base (0.00076 Btu/ft²sec-°R).

FIGURE A-51. EFFECT OF MODEL 1 WAKE ON DOUBLE WEDGE (MODEL 12) HEATING DISTRIBUTIONS AT MACH NUMBER 4.44



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FIGURE A-52. EFFECT OF MODEL 4 SHOCK IMPINGEMENT ON MODEL 1 HEATING DISTRIBUTIONS AT MACH NUMBER 2.49







FIGURE A-54. EFFECT OF MODEL 4 SHOCK IMPINGEMENT ON MODEL 1 HEATING DISTRIBUTIONS AT MACH NUMBER 4.44



- an average of clean plate heat-transfer coefficients obtained in the area covered by Model 2 base $(0.00249 \text{ Btu/ft}^2\text{sec-}^\circ\text{R})$. NOTE: Values presented are the ratio of local heat-transfer coefficient to
- FIGURE A-55. EFFECT OF MODEL 1 ON MODEL 2 HEATING DISTRIBUTIONS AT MACH NUMBER 2.49

FIGURE A-56. EFFECT OF MODEL 1 ON MODEL 2 HEATING DISTRIBUTIONS AT MACH NUMBER 3.51

an average of clean plate heat-transfer coefficients obtained in the area covered by Model 2 base (0.00140 Btu/ft²sec-°R) Values presented are the ratio of local heat-transfer coefficient to NOTE:







NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 2 base (0.00087 Btu/ft²sec-°R).



EFFECT OF MODEL 1 ON MODEL 4 HEATING DISTRIBUTIONS AT MACH NUMBER 2.49 FIGURE A-58.










- NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 5 base (0.00255 Btu/ft²sec-°R).
- FIGURE A-61. EFFECT OF MULTIPLE WAKES (MODELS 1 AND 2) ON MODEL 5 HEATING DISTRIBUTIONS AT MACH NUMBER 2.49



- NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 5 base (0.00135 Btu/ft²sec-°R).
- FIGURE A-62. EFFECT OF MULTIPLE WAKES (MODELS 1 AND 2) ON MODEL 5 HEATING DISTRIBUTIONS AT MACH NUMBER 3.51



NOTE: * indicates measurement located on opposite side.

- NOTE: Values presented are the ratio of local heat-transfer coefficient to an average of clean plate heat-transfer coefficients obtained in the area covered by Model 5 base (0.00085 Btu/ft²sec-°R).
- FIGURE A-63. EFFECT OF MULTIPLE WAKES (MODELS 1 AND 2) ON MODEL 5 HEATING DISTRIBUTIONS AT MACH NUMBER 4.44

P	•0.72	\$0.9J		2 .0.80	@0.72			7 00.75		H				ttio o cient fer same
	•0.74		tò't●	●0.85 ●0.85 ●0.85 ●0.8	\$ 0.74		0.77	#0,78 #0,79 #0,77 #0,79 #0.7	4 0.78 6			, , ,	te 0,92	sented are the ra -transfer coeffic plate heat-trans t obtained at the
• 0.78	•0.78 •0.78	•1.02	¢0.88 ¢0.92	@ 0.91 @ 0.89 @ 0.87	@0.71 @0.74 @0.74	\$0.88 \$0.87	€0.76 &0.77 €0.79 4	.76 e0.70 e0.78 e0.76 e	€0.78 €0.77		4 0.91			E: Values pre local heat the clean coefficien
			0.85 . 80.84	a a0.97 a 0.93	6.69 80.57 80.59 80.71	0.91 \$ \$0.88	f	G 8						LON
	•0.79		a 1.26 a 0.81	0.98 a1.21 a1.12	0.74 a0.83 a a	0.73 *0.52 1,45 *0.80	8 5 7 2 9 2 7 20 9 2 7 20 9 2 7 20 9		0.90'0.47 • • 1 • • 0.93 1.17	•0.88 140.80	0. 97			
			e +1.08 = = 1.45	0.92 0.79	•0.94 •0.82 •0.87 •	1 00.59 0.02 1.53	0.35 00.86	al.27 a0.54 00.00	4 8 8 8 0.71					
	•0.84				87 0.89 & 0.87	.12 0.85 0.58	•1.38	0.54	1 .57					

EFFECT OF 0.5 INCH HIGH HAT SECTIONS (MODELS 13 AND 14) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 12 FIGURE A-64.

	\$ \$0.67	8 0.86	@0.78 @0.79 @0.75			14 @ 0.73 @ 0.70 @ 0.59						e the ratio (coefficient t-transfer I at the same
	•0.7	€1.06	€7.0€ €0.71		@0.78	@0.70@0.72@0.	@0.72 @			8	¢0.95	resented ar at-transfer n plate hea ent obtained
.77	.81 •0.81	.90 •0.94	.94 ©0.91 ©0.85 .70 ©0.71 ©0.71	@0.85 @0.86	.77 @0.77 @0.79	2 @ @ 0,73 @ 0.73 @	71 @0.72		@ 0.91			: Values p local hee the clean coefficio location
0	0	28.00	e1.03 e0.95 e0	• • 0.93) •	● 0.7.) ®					NOTE
	•0.81		e 1.23 e 1.23	0.44 \[1.66 & 0.78	22 22 22 23 27 27 23 27 23 26 26 26 26 26 26 26 26 26 26 26 26 26			70	@1.03			
		€ 88 80 1.91 1.91 1.91 1.91	*0.*0 *0.*0 *0.*90	©0.92 ¢[1,76 ©0.65] ©0	0.73 0.13	42 42 42 42 42 42 42 42 42 42	2 · 0 • 2 · 0 •					
		9 5	9.0° 9.0° 9.0° 9.0° 9.0°	\$0.96 \$0	0.2	80.47 %1.38 %0 80.45 %1 % %0	\$29 \$40 \$20 \$0 \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$ \$				9	
	•0.86		00' I* 86' 0* 28' 0* 68' 0*	82'00 	11 19.1 1		@1.89 @1.13					

EFFECT OF 0.5 INCH HIGH HAT SECTIONS (MODELS 13 AND 14) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 12 FIGURE A-65.

•0.73	•0.75 •0.75 •0.72 •0.71 •0.71 •0.71 •0.71 •0.71 •0.72 •0.71 •	2.65 1.78 1.38 0.82 0.77 0.77 0.01	• • • • • • • • • • • • • • • • •	12.31 1.00 11.36 1.1 0.67 0.67 0.67 0.67 0.67 0.60 0.59 0.61 0.67 0.67 0.67 0.67 0.67 0.67 0.51 0.51 0.51 0.51	.84 0.95 0.15 0.15 0.16 0.77 0.46 0.77 0.49 0.47 1.57 0.72 0.90 0.86 0.72 0.72 0.72 0.12	0.78 0.70 11 1<			•0.97	•1.03	NOTE: Values presented are the ratio of local heat-transfer coefficient to the clean plate heat-transfer coefficient obtained at the same location.
	6 0.83				0.96 0.84 0.95 0.94 1.49 0.80 0.80 0.45	0.84 [0.78 0.70 =1 84 []	e e e e e e e e e e e e e e e e e e e				

FIGURE A-66. EFFECT OF 0.5 INCH HIGH HAT SECTIONS (MODELS 13 AND 14) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 12



EFFECT OF 0.75 AND 1.0 INCH HIGH HAT SECTIONS (MODELS 15 AND 16) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 2.49 - CONFIGURATION 13 FIGURE A-67.



EFFECT OF 0.75 AND 1.0 INCH HIGH HAT SECTIONS (MODELS 15 AND 16) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 3.51 - CONFIGURATION 13 FIGURE A-68.



EFFECT OF 0.75 AND 1.0 INCH HIGH HAT SECTIONS (MODELS 15 AND 16) ON PLATE HEATING DISTRIBUTION AT MACH NUMBER 4.44 - CONFIGURATION 13 FIGURE A-69.

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