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STUDY OF STABILITY AND CONTROL MOMENT GYRO
WOBBLE DAMPING OF
FLEXIBLE, SPINNING SPACE STATIONS

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STUDY OF STABILITY AND CONTROL MOMENT GYRO
WOBBLE DAMPING OF
FLEXIBLE, SPINNING SPACE STATIONS

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FOREWORD

This Final Report is submitted to the NASA Manned Spacecraft Center in partial fulfillment of Contract NAS 9-11991. Presented herein are the results of the technical investigations along with conclusions and recommendations for further action in the area of stabilization and control of flexible, spinning space stations.

This report has been segmented into three parts. PART I, which can be considered an Executive Summary, presents an introduction, a concise summary of significant results, and the principal conclusions and recommendations. PART II presents a description of the analyses along with an in-depth discussion of the results. PART III is a User's Guide for the digital computer program that simulates the flexible, spinning space station. The development and submittal of this computer program is a contractual requirement.

The study program began on June 28, 1971. Mr. K. Lindsay of the Guidance and Control Division at NASA/MSC was Contract Technical Monitor. The overall study was directed and coordinated by Mr. H. Berman, Project Engineer. Mr. Berman also led the control analysis activities with aid from Mr. W. Holmer. Dr. J. Markowitz was responsible for the derivation of the dynamic equations of motion and the modal analysis. Dr. F. Austin provided valuable advice in various aspects of the program.

Other contributors to the successful completion of this task were: George Zetkov of the Guidance and Control Section, Arthur Schloth of Loads and Dynamics, and Howard Flicker of Grumman Data Systems.

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PART IIntroduction, Summary, and Conclusions1.0 Introduction

The open-loop (uncontrolled) stability of the space station and the control moment gyro (CMG) requirements for damping the wobble motion of the space station can be influenced significantly by structural flexibility and vehicle mass distribution. The purpose of this study is to provide an early evaluation of open-loop stability which will allow for sufficient lead time to modify, if necessary, the basic structural design of the space station. If the uncontrolled vehicle is unstable and if changes cannot be made to render the uncontrolled vehicle stable, then CMG reliability would assume vastly greater importance. In addition, early awareness of the CMG system requirements will provide sufficient lead time for hardware development. As a result, the following study objectives are formulated:

- o Determine the effects of structural flexibility on the uncontrolled stability of two dual-spin space station configurations.
- o Investigate CMG wobble damping with an emphasis on control interaction with the structure.
- o Develop and deliver to MSC a digital simulation of the flexible, spinning space station.

The newly developed simulation will be used as an analysis tool to satisfy the first two study objectives; also, it will be available to MSC personnel for future studies.

NASA/MSC has selected for investigation two typical modular space station configurations with extremely different mass distributions. Both configurations consist of a spinning member (a rotor) rotating about a near stationary member (the hub).

In order to accomplish the above outlined objectives for the two general vehicles chosen for investigation, the following study approach is adopted:

- o Establish wobble damping requirements, and the level of wobble induced disturbances.
- o Formulate the general flexible-body equations of motion and develop the corresponding digital simulation.
- o Perform preliminary open-loop stability analyses and CMG wobble damping analyses using digital computer programs developed during previous Grumman studies.
- o Use the newly developed simulation to evaluate open-loop stability and analyze sensitivity to variations in the space station structural properties.
- o Use the newly developed simulation to evaluate the performance of the CMG wobble damping concept developed during a previous Grumman study.

2.0

Summary of Results

The two space station configurations selected for investigation are referred to as the "T" and "Y." The "T" and "Y" configurations were chosen so that each general class of rotating satellites with extremely different mass distributions is represented; specifically, "T" is a "Min I" vehicle (the inertia about the rotor spin axis is smaller than either transverse inertia), while "Y" is "Max I" (the inertia about the rotor spin axis is greater than either transverse inertia).

The majority of the analyses were performed using two simulations: the Simplified Flexible Body Simulation (developed during a previous study for NASA/MSFC) and the General Flexible Body - Stage II simulation (developed during this study). The Simplified Flexible Body simulation idealizes the vehicle as a rigid rotor connected to a rigid hub by a flexible bearing. In the new simulation, the structure is idealized by segmenting the vehicle into several flexible and rigid sections. The sources of structural flexibility considered are:

- o The bearing, connecting the rotor to the hub
- o The long, thin walled sections that comprise the rotor arms and hub
- o The space station-shuttle attachment point.

Much of the previous stability analyses of flexible rotating satellites are based upon many simplifying assumptions, and non-rigorous energy sink methods. Hence, results of the present analyses, using high fidelity structural simulations, enhances the previous knowledge learned from the earlier studies. Using the Simplified Flexible Body Simulation, which physically represents a vehicle with a highly flexible bearing compared to relatively stiff rotor arms and hub, the following uncontrolled stability information is uncovered:

- o "Y" is always stable
- o "T" is unstable with damping in the bearing outer race
- o A stability boundary exists for the "T" configuration with bearing inner race damping.

Using a range of stiffness values for rotor arm and hub flexibility in conjunction with the newly developed simulation, the following occurred:

- o The uncontrolled "Y" configuration is stable even with a flexibly attached shuttle. It should be noted that a "Y" station with a shuttle is a "Min I" vehicle, where stability is always questionable.
- o The uncontrolled "T" configuration is marginally stable. It appears that there is sufficient energy dissipation in the non-rotating portion of the vehicle to over-power the unstable influence of energy dissipation in the rotor. The latter is reaffirmed by demonstrating an instability for an uncontrolled "T" with a rigid hub.

The wobble damping requirements are based upon achieving a $10^{-5}g$ environment throughout the hub within a specified time from the initiation of the wobble induced disturbance. Although CMG wobble damping is investigated in response to shuttle docking disturbances, CMG requirements will not be based upon the relatively infrequent dockings; rather, successful damping of the wobble induced by typical internal mass motion will determine the CMG requirements.

CMG wobble damping is investigated using a unique CMG configuration that was developed during Grumman's previous study on space-base wobble damping. The CMG control technique employs a single-gimbal CMG with an associated control strategy that forces the wheel spin axis to lag the wobble vector by 90 degrees in inertial space. The interaction of the CMG wobble damping scheme with the flexible space station is evaluated using the General Flexible Body Simulation - Stage II. The results of the investigations are summarized below.

Little modification to the CMG system is required when operating with the "Y" station. A low pass filter with a time constant of 2 seconds is added to decrease the actuator duty cycle. The CMG system required the following modifications in order to perform successfully with the "T" station:

- o A negative limit on gimbal rate to prevent a control instability.
- o A low pass filter with a time constant of 10 seconds is needed to effectively decrease the actuator duty cycle.

- o A bias on the gimbal angle command is required in order to minimize the CMG tracking error introduced by the filter.
- o The criterion for stopping active CMG damping is raised in order to avoid interacting with the structural modes at low wobble amplitudes.

In addition to the above noted CMG system modifications required for successful wobble damping of the "T" station, the "T" configuration, as compared with "Y", will in general require a significantly larger CMG and greater torquer motor output power. Specifically, in response to a typical profile of internal mass motion the following requirements are established:

Configuration	Wheel Size Angular Momentum (ft-lb-sec)	Maximum, Average Power Output (watts)
"Y"	400	0.009
"T"	12,000	5.15

The development of the General Flexible Body - Stage II simulation was a contractual requirement. A user's guide for this simulation has been prepared and is presented in Part III of this report.

3.0 Conclusions and Recommendations

There should be no concern over the uncontrolled vehicle stability of a "Y" type space station. The results of all investigations indicate that the vehicle is stable regardless of the amount or location of energy dissipative structural material. In fact, the time duration for wobble convergence will decrease as the amount of structural flexibility is increased; whether the increase in flexibility is in the rotating portion of the vehicle or non-rotating portion, is unimportant. In contrast, the uncontrolled vehicle stability of a "T" type space station is extremely sensitive to the amount and location of energy dissipative material. For example, it is shown that an increase in structural flexibility in the spinning portion has a de-stabilizing influence while an increase in the flexibility of the non-rotating section has a stabilizing effect. Therefore, to enhance the chances for uncontrolled vehicle stability, additional stiffening members should be added to the "T" rotor. It should be noted that the nominal "T" configuration is marginally stable. However, the flexible parameters assumed as nominal represent preliminary estimates. As more structural data is made available, the uncontrolled stability of the "T" configuration should be reaffirmed.

The CMG requirements associated with the "T" configuration are extremely unattractive. Furthermore, to satisfy the wheel size requirement with one CMG would be beyond the "state-of-the-art." In contrast, the CMG requirements associated with the "Y" station can be satisfied without developing additional CMG technology.

Due to the stabilization and control problems associated with the "T" space station, a configuration of this type should be avoided, assuming the option is available. If the option is unrealistic, then early development of a highly reliable CMG system with a very large angular momentum capacity is recommended.

Future Studies

The work accomplished as part of this study solves just a small part of the complex space station control problem. As a result, the following directly related areas of study are recommended for future investigation.

It has previously been established that along with a CMG wobble damping system, the space station will require the following additional spin vector control systems:

- o Mass Balancing System
- o RCS Attitude Control for Pointing
- o Spin Rate Control

Aside from it being important to study the interaction of each control system with the flexible structure and uncover any possible problems, it is equally important to study the integrated spin vector control system and uncover any problems associated with the interaction between control systems.

It is noted that a complex spin vector control system of this type would be advancing present technology and should be studied as early as possible.

In addition to studies in the space station technology area, the simulation that has just been developed as part of this study can be used to investigate the flexible body dynamics of other space vehicles. Specifically, by manipulating the input data, the program can be used to simulate a variety of vehicles with rotating and non-rotating sections, whether it be one rigid body with six degrees of freedom to fifteen flexibly connected bodies with eighty-four degrees of freedom. With minor modifications to the program, the simulation can be made to represent dual spin satellites with an arbitrary number of rotor arms, as well as a high fidelity simulation of a flexible Shuttle Orbiter.

4.0 Physical Vehicle Properties

4.1 Vehicle Configurations

NASA/MSC has selected for investigation two typical modular space station configurations. Both configurations consist of a spinning member (the rotor) rotating about a near stationary member (the hub). The end module of each rotor arm serves as living quarters within which an artificial gravity environment is experienced for the life of the station. Inside the hub, where most of the scientific experiments are performed, a near zero-g environment is provided. The two configurations chosen for study are the "T" and "Y" configurations, and are illustrated in Figures 4.1 and 4.2, respectively. For reference purposes, these two configurations can be found on official MSC drawings SAY44101266, and SAY44101264.

The "T" and "Y" configurations were chosen so that each general class of rotating satellites is represented; specifically, "T" is a "Min I" vehicle (the inertia about the rotor spin axis is smaller than either transverse inertia), while "Y" is "Max I" (the inertia about the rotor spin axis is greater than either transverse inertia). The "Y" configuration may be considered an outgrowth from "T", in that the "T" rotor is comprised of two arms, 180° apart, while the "Y" has a three armed rotor, 120° between arms.

In order to study the dynamics of a flexible space station, the structure is idealized by segmenting the vehicle into several flexible and rigid sections.

The sources of structural flexibility considered are:

1. The bearing, connecting the rotor to the hub
2. The long, thin-walled sections that comprise the rotor arms and hub.
3. The shuttle-orbiter attachment point.

The structural properties of each flexible member are detailed in Section 4.3. Section 4.2 presents their mass properties.

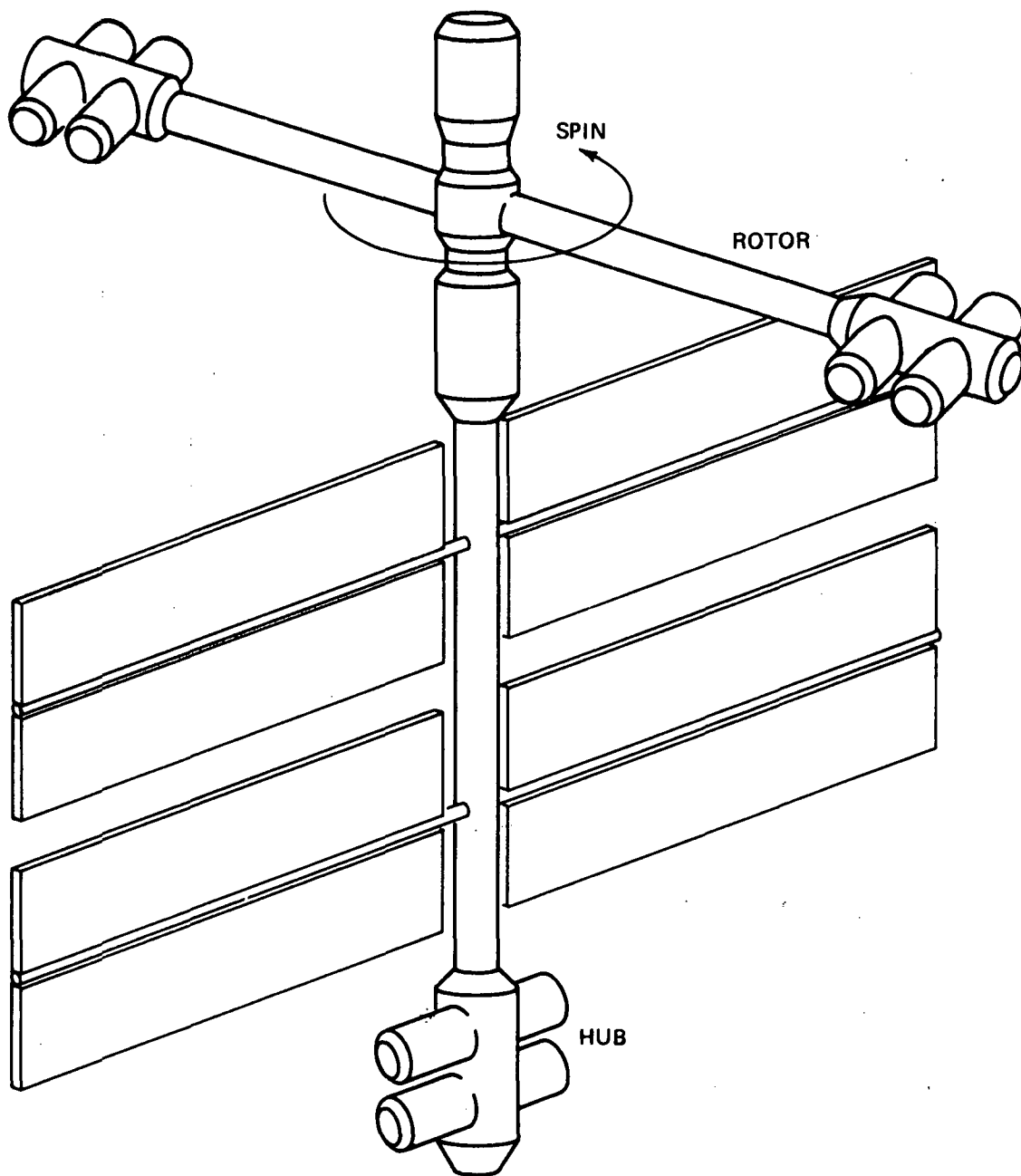


Figure 4-1. T Space Station

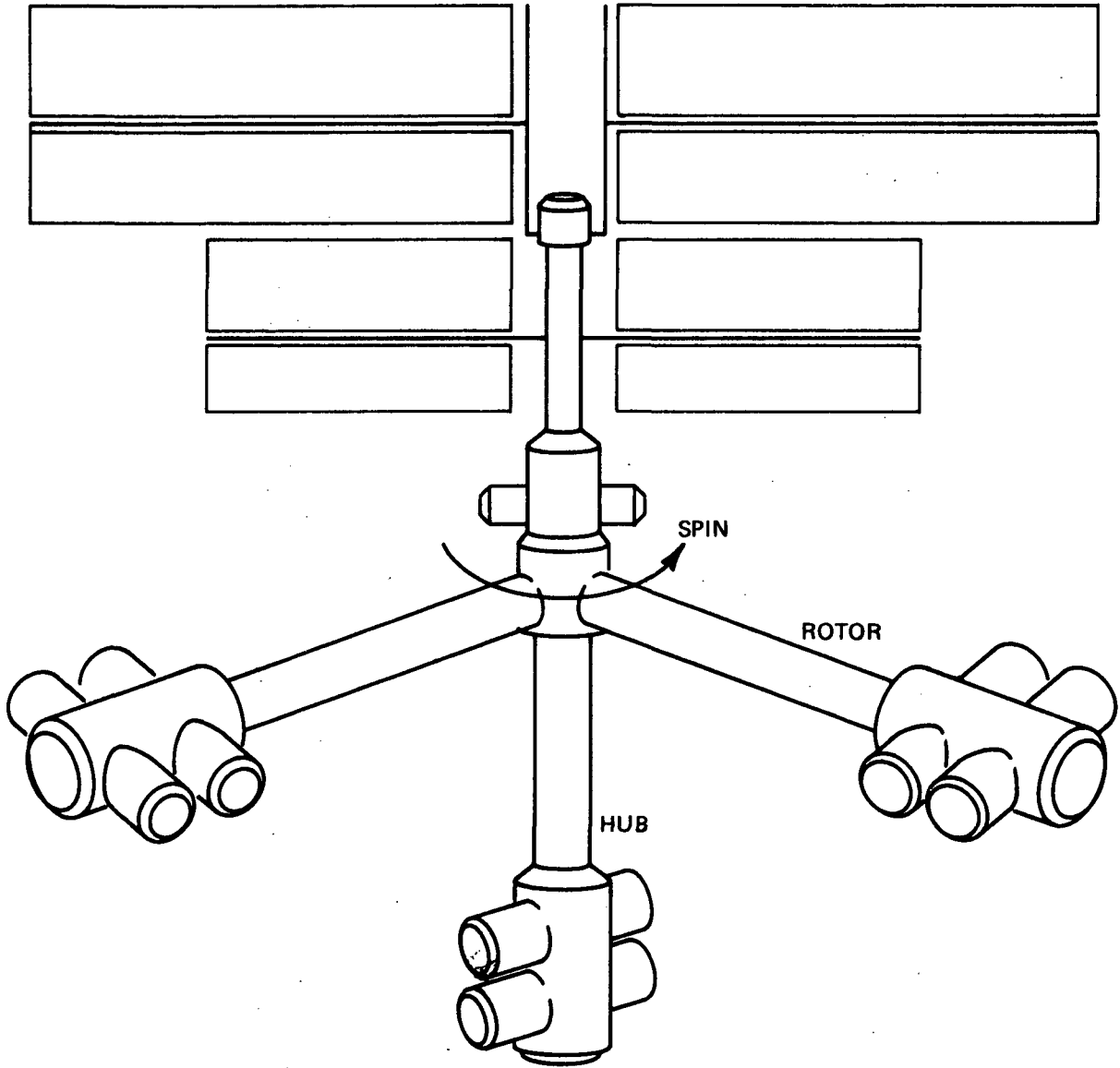


Figure 4-2. Y Space Station

4.2 Mass Properties

Mass property data for the main bodies (rotor and hub) of each configuration was supplied by MSC in Reference 1. A duplicate table, obtained from Reference 1, is shown as Table 4.1 for completeness.

The vehicle idealization used for the dynamic analyses of the flexible space station is a vehicle segmented into several lumped mass sections as shown in Figures 4.3 and 4.4. Specifically, the idealized vehicle is comprised of:

- ° 3-lumped masses per rotor arm
- ° 1-rotor ring
- ° 3-lumped masses for the flexible hub section
- ° 1-rigid hub section

Mass property data for each lumped mass was developed based upon modular, or building block, data provided in Reference 1. Figures 4.3 and 4.4 contain tabulated mass property data for each lumped mass section. The mass property data shown is in the proper form for direct use in the simulation that was developed during this study.

The analyses that included a shuttle orbiter attached to the station used orbiter mass properties corresponding to a delta wing-25K payload orbiter as obtained from Reference 2, and listed in Table 4.2.

TABLE 4.1 MASS PROPERTY DATA

CONFIGURATION		WEIGHT (lbs x 10 ⁻³)	C.M.-(FT) ⁽¹⁾			INERTIAS (SLUG-FT ² X 10 ⁻⁸) ⁽²⁾		
			X	Y	Z	Ixx	Iyy	Izz
T	Rotor	285	0	0	0	.302	1.127	.889
	Hub	184	0	0	140.7	.637	.661	.025
	Total	469	0	0	55.1	.939	1.788	.914
Y	Rotor	431	0	0	0	.745	.745	1.330
	Hub	156	0	0	- 58.3	.220	.256	.037
	Total	587	0	0	- 15.5	.965	1.001	1.33

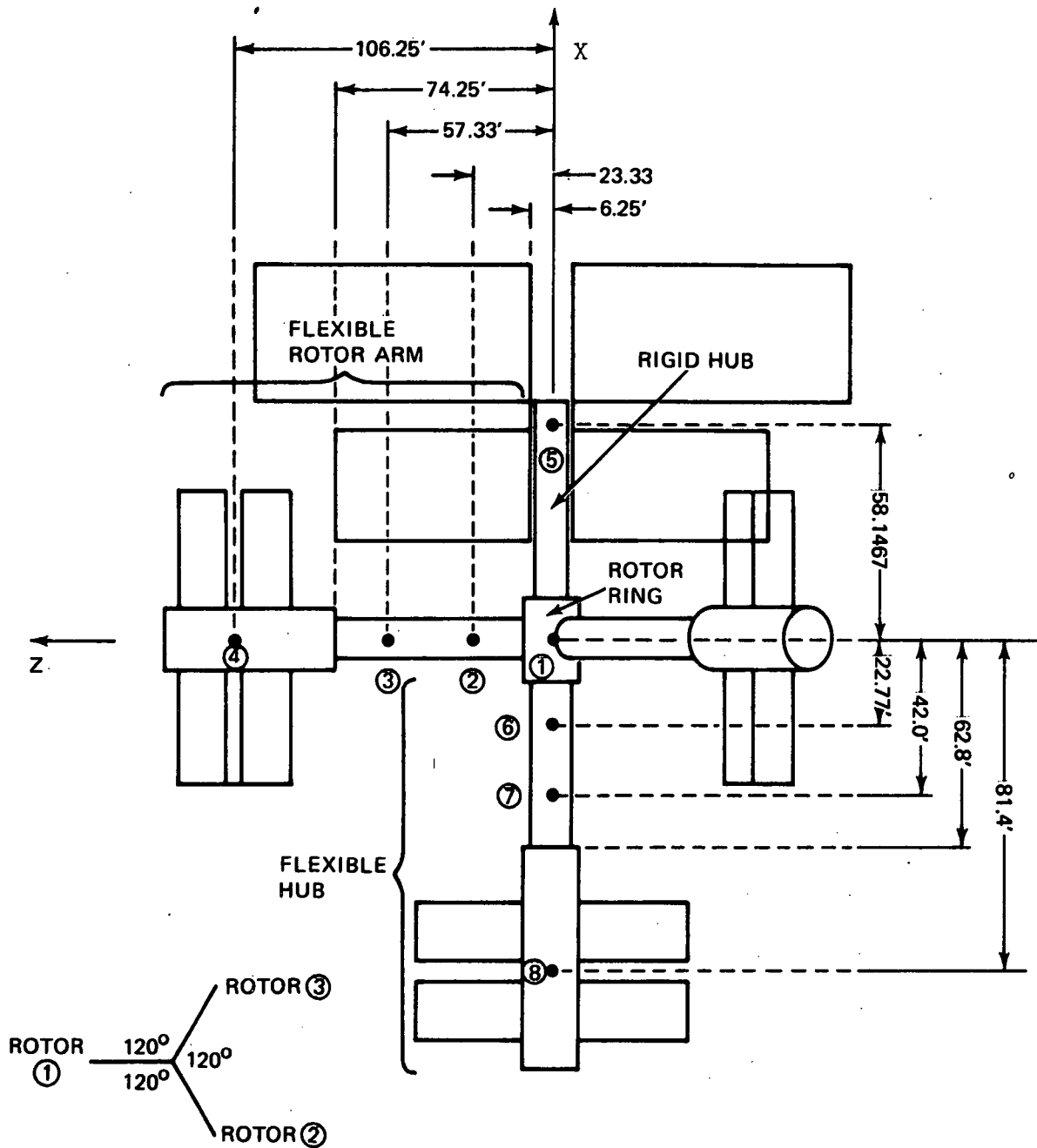
NOTES: 1 The center of the xyz coordinate system is in the rotor plane.

2 Inertias are measured relative to the total CM. Z axis is spin

TABLE 4.2 SHUTTLE ORBITER - DELTA WING - 25 K PAYLOAD MASS PROPERTIES

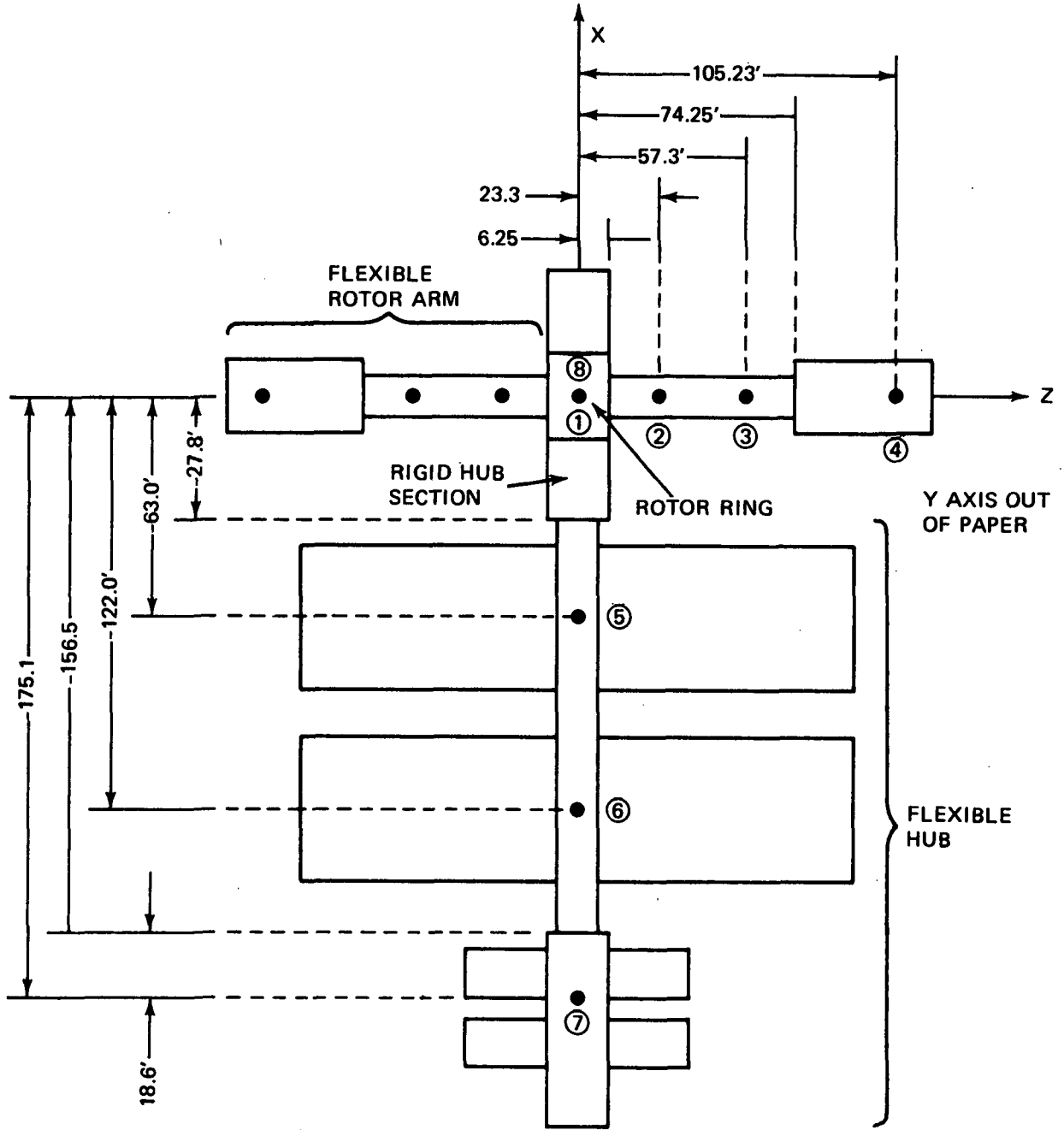
Weight (lb)	x ^{sta*} (inches)	Moment of Inertia slug-ft ² x 10 ⁻⁶		
		Ixx	Iyy	Izz
316940.	1521.8	2.725	19.883	21.495

*Measured from nose of orbiter



Element Number	Element Type	Ixx (Slug-Ft ²)	Iyy (Slug-Ft ²)	Izz (Slug-Ft ²)	Weight (Lbs)
1	rotor ring	12,692.0	32,781.4	32,781.4	19,000
2	rotor	22,731.1	22,731.3	2,083.	7,250
3	rotor	22,731.3	22,731.3	2,083.	7,250
4	rotor	469,481.85	2,018,651.07	1,642,578.22	123,000
5	rigid hub section	2,007,596.1	2,686,897.86	682,636.59	23,000
6	hub	1,416.43	5,757.926	5,757.926	4,500
7	hub	1,005.43	3,120.47	3,120.47	3,500
8	hub	1,644,085.22	2,034,929.53	485,759.91	125,000

FIGURE 4.3 Y CONFIGURATION IDEALIZATION/MASS PROPERTIES



Element Number	Element Type	I_{xx} (Slug-Ft ²)	I_{yy} (Slug-Ft ²)	I_{zz} (Slug-Ft ²)	Weight (Lbs)
1	rotor ring	11,296	15,586	15,586	15,000
2	rotor	22,731.3	22,731.3	2,083	7,250
3	rotor	22,731.3	22,731.3	2,083	7,250
4	rotor	2,034,929.53	485,759.91	1,644,085.22	125,000
5	hub	1,735,329.45	2,012,994.74	292,228.82	23,886.03
6	hub	1,733,430.57	1,829,283.28	98,517.36	17,275.74
7	hub	1,644,085.2	2,034,929.53	485,759.91	125,000.
8	rigid hub section	9,007.36	146,270.93	146,270.93	16,067.25

FIGURE 4.4 T CONFIGURATION IDEALIZATION/MASS PROPERTIES

4.3 Structural Properties

4.3.1 Rotor and Hub Extension Arms

The extension arms for both the rotor and hub are constructed from 7 foot outer diameter cylindrical sections of Aluminum alloy (2219-T87). Moments of inertia (I and J) and elastic moduli (E and G) for these sections were provided by NASA and are listed below:

- ° $I = 13,446 \text{ in}^4$
- ° $J = 26,892 \text{ in}^4$
- ° $G = 4 \times 10^6 \text{ psi}$
- ° $E = 10^7 \text{ psi}$

Based upon the above polar moment of inertia (J) the computed cross-sectional area is:

- ° $A = 15.8 \text{ in}^2$

The computed stiffnesses are as follows:

- ° Bending $EI = 9.34 \times 10^8 \text{ lb-ft}^2$
- ° Torsion $GJ = 7.47 \times 10^8 \text{ lb-ft}^2$

The above values are considered baseline, lower, stiffness limits. For an upper bound which will also be considered nominal stiffness, the above values were doubled. The rationale for doubling the calculated stiffnesses and using the result for nominal is that stiffening members were excluded from the original calculations.

The above stiffnesses in combination with the geometry of the extension arms are used as input data for the development of a stiffness matrix for both the rotor and hub (refer to Appendix A for the definition of the stiffness matrices). The resulting stiffness matrix is combined with the mode shapes, frequencies, and the percentage of critical damping (as shown in Appendix A), to obtain the damping matrix. One percent structural damping in the extension arms is considered nominal and used for the majority of the analyses.

4.3.2 Rotor-Hub Bearing

The rotor is connected to the hub through a massive bearing that no doubt will be highly flexible. The assumed bearing model is the same as in Reference 3, and is described in section 6.3 of this report.

A reasonable estimate for the elastic characteristics of the bearing can be obtained by assuming a structural natural frequency (f_{NB}) of 0.25 cps. By applying,

$$\frac{K_B}{J_R} = (2\pi f_{NB})^2 \quad (1) \quad (4-1)$$

where

K_B = Bearing stiffness

J_R = Maximum transverse inertia of the rotor,

the following estimate for nominal bearing stiffness is obtained,

$$K_B = 2.1156 \times 10^8 \frac{\text{ft-lb}}{\text{rad}}$$

Most likely, as part of the bearing fabrication process the rotor arms will either be riveted, bolted, or welded to the bearing outer race. The material at these attachment points will be highly dissipative. Therefore, it is assumed that 10% of critical damping is considered nominal damping for the bearing. By applying,

$$E_B = 2\gamma \sqrt{K_B J_R} \quad (1) \quad (4-2)$$

where

E_B = Damping coefficient

γ = Percent critical damping

The following estimate for nominal bearing damping is obtained,

$$E_B = 2.695 \times 10^7 \frac{\text{ft-lb}}{\text{rad/sec}}$$

1 The derivation of the elastic parameters for the bearing is based upon a simple model where the hub is fixed and the relative angle between the rotor and hub (θ) is given by,

$$\ddot{\theta} + \frac{E_B}{J_R} \dot{\theta} + \frac{K_B}{J_R} \theta = 0$$

Although the above values are presently thought to be nominal they are subject to change as more structural details become available. It is for this reason that a parametric study was performed to determine the variation in vehicle performance as a function of the bearing parameters (refer to section 8.2).

It is noted that nominal bearing damping, as described above, locates dissipative material in the rotating section of the vehicle. As part of this study there were investigations performed to determine any performance differences associated with locating damping material in the non-rotating section of the bearing (refer to section 8.2).

4.3.3 Shuttle Docking Mechanism

During the analyses that included a Shuttle orbiter attached to the station, a rigid orbiter was assumed to be flexibly attached to the rigid hub section. Nominal values for the elastic coefficients associated with the docking mechanism are obtained in the same manner as the bearing coefficients were obtained. However, the natural frequency for the shuttle connection is assumed to be 0.5 cps, and the structural damping is assumed to be nominally at 1%. The resulting values for the stiffness and damping coefficients are:

$$K_H = 2.119 \times 10^8 \text{ ft-lb/rad}$$

$$E_H = 1.35 \times 10^6 \text{ ft-lb/rad/sec}$$

4.4 Vehicle Spin

A nominal rotor spin rate equal to 2.5 RPM (.262 rad/sec) is used throughout this study, as per an MSC recommendation.

5.0 Wobble Damping Requirements and Disturbances

5.1 Damping Requirements

The desired motion of the space station is pure spin. Any torque free oscillatory motion of the vehicle in inertial space other than pure spin is commonly referred to as WOBBLE. The function of the CMG will be to remove the wobble and in turn restore the vehicle to a pure spin state. Wobble damping requirements consists of an acceptable wobble level along with the time required to damp within the acceptable range. The term wobble level is used synonymously with wobble rate, or the oscillatory component of angular rate viewed on axes perpendicular to the nominal spin axis (i.e. transverse axes).

After reviewing the available control system requirements (as stated in References 4 and 5) and the physiological requirements (established in Reference 6) it is determined that the experimental requirement for a $10^{-5}g$ environment imposes the most stringent restriction on the wobble level. Table 5.1 contains a summary of the Navigation and Control requirements for an artificial gravity space station.

In order to obtain a specification for the maximum acceptable wobble level (ω_r) based upon the $10^{-5}g$ requirement, the following inequality is solved:

$$|r \dot{\omega}_T| \leq 10^{-5}g \quad (5-1)$$

Where, r = maximum distance from CM of total space station to the farthest point in the hub (feet)

$\dot{\omega}_T$ = transverse angular acceleration (rad/sec)

It should be noted that the $\omega_T^2 r$ contribution to the total linear acceleration is negligible (i.e., $|\dot{\omega}_T| \gg \omega_T^2$) during the periods of interest characterized by low amplitude wobble. Making the appropriate substitutions in (5-1) for $\dot{\omega}_T$ in terms of the wobble rate, the spin rate (P) and the inertia properties, and in turn solving for the wobble amplitude yields,

TABLE 5-1

NAVIGATION AND CONTROL REQUIREMENTS

N and C Requirement	Source	Reference
Maintain Attitude Hold and Rate Stabilization for 30 days: ±0.1 deg, ±.003 deg/sec	MDAC	(4)
Wobble Limit Art-G Experiment: ±0.1 deg/sec	NR	(4)
General Experimental Requirements: Acceleration Level <.01 g 100% of time <.00001 g 95% of time <.0001 g 5% of time	--	(4)
Physiological: Ang. Acc. <0.15 deg/sec ² Wobble Ang. <15 deg Vibration <.005 in. displacement <.01 g pk acceleration	TRW	(6)
Routine Operations Attitude Hold: ±0.25° Rate Stability: ±0.05°/sec	--	(5)

$$\omega_r = \left\{ \begin{array}{l} \alpha \Omega \text{ for } \alpha > 1 \\ \Omega \text{ for } \alpha \leq 1 \end{array} \right\} = \text{Maximum Acceptable Wobble Level (rad/sec)}$$

where,

$$\Omega = \frac{10^{-5} (32.2)}{[\alpha \pm \lambda] \text{pr}} \left(\frac{\text{rad}}{\text{sec}} \right) \quad \text{Note: 1. Use minus sign for Min I and plus sign for Max I.}$$

$$\alpha = \left[\frac{(I_x - I_z) I_z}{(I_x - I_y) I_y} \right]^{\frac{1}{2}}$$

2. The x axis is the spin axis.

$$\lambda = \left[\frac{(I_x - I_z) (I_x - I_y)}{I_z I_y} \right]^{\frac{1}{2}}$$

The calculated maximum allowable wobble levels corresponding to the space station configurations studied are presented in Table 5.2.

There is presently no firm requirement for the rate of wobble decay. MSC has been using a 60 sec time constant (τ) for CMG sizing purposes (Reference 7), which is considered a reasonable requirement and will, therefore, be adopted for this study with the following slight modification. Since the particular CMG law to be investigated will exhibit a linear decay, rather than exponential, a total decay time requirement of 180 sec is specified, which is equivalent to 3τ .

TABLE 5.2

WOBBLE RATE SPECIFICATION

Configuration	r (feet)	α	λ	Maximum Allowable Wobble ω_r (rad/sec)
"T"	152.9	5.852	.1636	.824 x 10 ⁻⁵
"T" with Shuttle	247.8	1.067	0.800	1.98 x 10 ⁻⁵
"Y"	84.5	.967	.3526	1.1 x 10 ⁻⁵
"Y" with Shuttle	151.4	1.030	.623	2.05 x 10 ⁻⁵

5.2

Disturbances

Deviations from the desired pure spin motion of the space station will be induced by both internal and external disturbances. The three most significant sources of disturbance torques are:

- o Internal Mass Motion
- o Shuttle Docking
- o Environmental

Although momentum exchange devices are applicable for counteracting the oscillatory portion of environmental disturbances, the effect of these disturbances is a precession of the vehicle's momentum vector, and thus is an attitude hold problem rather than a wobble damping problem. As a result, the study of environmental disturbances is considered beyond the scope of these investigations.

The result of a shuttle impact on the space station will be a displacement of the station in addition to a wobble motion about the new displaced position. RCS jets can be used to restore the vehicle to its original position while a CMG system can be used to damp the wobble motion. It should be noted that the use of CMG's for wobble damping decreases the load on the RCS system and will result in less propellant consumption.

Unbalances in the rotor that displace the CMG from the spin axis will result in unacceptable motion of the station which cannot be compensated for by CMG's, thus a mass-balancing system is required. In addition to introducing large cross-products of inertia, which in a sense will create a new spin axis, rotor unbalances will cause the hub to translate in space resulting in large acceleration levels in a preferred near zero-g environment. However, internal mass motion will also induce a wobble motion superimposed on these other effects. It is only the wobble motion that is considered during the course of this study.

Shuttle Docking

The total angular momentum imparted to the station by a planar shuttle docking is given by,

$$\Delta h_{\text{DOCK}} = I_s \omega_s + m_s v_s l_s, \quad \text{or} \quad (5-3)$$

$$\Delta h_{\text{DOCK}} = I_s \omega_s + M_x [l_D (v_A \cos \theta_M - v_L \sin \theta_M) + l_{\text{CM}} (v_A \sin \theta_M + v_L \cos \theta_M)] \quad (5-4)$$

Where all variables are defined in Figure 5-1. Applying the maximum docking parameters and docking port contact requirements, as listed in Figure 5-1 (Reference 4: Table 6.1 - Operational Design, and Section 6.2.1.6), results in the following worst case (or 3 sigma) docking disturbance:

$$\text{"T"} - \Delta h_{\text{DOCK}} = 590,163 \text{ ft-lb-sec}$$

$$\text{"Y"} - \Delta h_{\text{DOCK}} = 613,760 \text{ ft-lb-sec}$$

The corresponding wobble amplitude for the shuttle/station combination will be $.0013 \frac{\text{rad}}{\text{sec}}$, and $.00176 \frac{\text{rad}}{\text{sec}}$ for the "T" and "Y" configurations, respectively. The CMG analyses, presented in Section 9, considers only a nominal (or 1 sigma) docking disturbance.

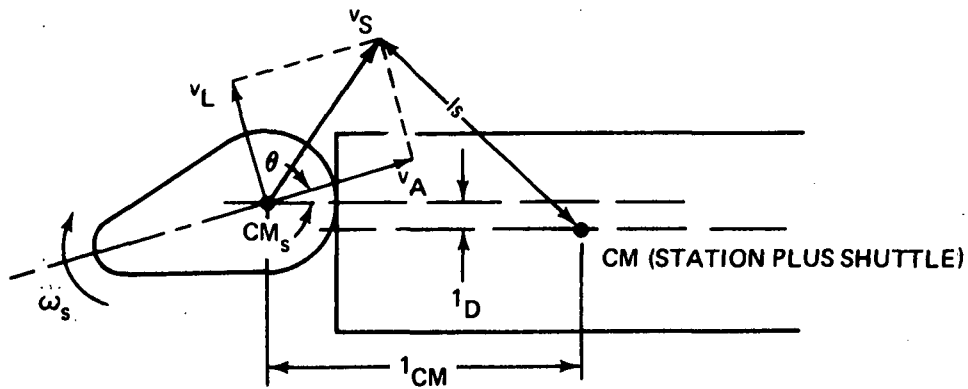
Internal Mass Motion

The wobble amplitude induced by internal mass motion will be a function of the size of the moving mass and its motion profile. MSC has performed a rigid body analysis for CMG sizing purposes using a particular moving mass profile (Reference 7) that induced wobble amplitudes as listed in Table 5-3.

TABLE 5-3

WOBBLE INDUCE BY MASS MOTION

<u>Configuration</u>	<u>Transvers Rate (rad/sec)</u>
"T"	0.4869×10^{-2}
"Y"	0.262×10^{-3}



MAXIMUM DOCKING PARAMETERS (REFERENCE 4):

$l_D = 5$ INCHES

$\theta_M = 4$ DEG

$\omega_s = 0.5$ DEG/SEC

$v_A = 1.0$ FPS

$v_L = 0.25$ FPS

VEHICLE PARAMETERS (SECTION 4)

SHUTTLE INERTIA = $I_s = 21.5 \times 10^6$ SLUG - FT²

CONFIGURATION	l_{CM} (FEET)	I_{TOTAL} SHUTTLE/STATION (SLUG - FT ²)
T	126.5	4.39×10^8
Y	133.99	3.49×10^8

1 SIGMA WOBBLE AMPLITUDE

CONFIGURATION	$\frac{(\Delta h_{DOCK})_{1\sigma} \text{ (RAD/SEC)}}{I_{TOTAL}}$
T	0.43×10^{-3}
Y	0.59×10^{-3}

Figure 5-1. Shuttle Docking

These same wobble amplitudes are used as initial conditions for the flexible body analyses performed as part of the present study. The rationale for using identical initial conditions is that the angular momentum transferred to a rigid vehicle by internal mass motion will be approximately equal to the momentum transferred to a flexible vehicle.

6.0 Vehicle Math Model

6.1 Equations of Motion

This section contains the equations of motion of the flexible space station. The analytical formulation of these equations was based upon a model consisting of the following components (See Figure 6-1):

- A rigid section of hub with 6 degrees of freedom (6 d.o.f.)
- A flexible section of hub containing p masses interconnected by massless structure ($6p$ d.o.f.)
- A rigid rotor ring pinned to the rigid section of hub by means of a flexible bearing (3 d.o.f.)
- q flexible rotor arms, each with s masses interconnected by massless structure ($6qs$ d.o.f.)
- A rigid shuttle pinned to the rigid section of hub by means of a flexible docking mechanism (3 d.o.f.)

The equations of motion are presented in Equations 6-1 through 6-8; associated relationships required for the formation of these equations are presented in Equations 6-9 through 6-31. Table 6-1 contains the definitions for all notations. These equations were linearized only with respect to those coordinates which describe the flexible motion of the vehicle; all other nonlinear effects were retained.¹ Mass properties of all bodies were completely general; that is, no assumptions were made regarding symmetries, position of mass centers, etc. Control torques were included on all masses.

Equation 6-43 contains the matrix equations of motion for the two vehicle configurations used in the simulation study²; the characteristics of these vehicles are as follows:

-
- 1 - Linearization of the flexible motion was not total; that is, nonlinear terms were eliminated only if significant simplification resulted.
 - 2 - Actually, Equation 6-43 can be used for any vehicle, with or without shuttle, for which $p = 3$ and $q = 2$ or 3 .

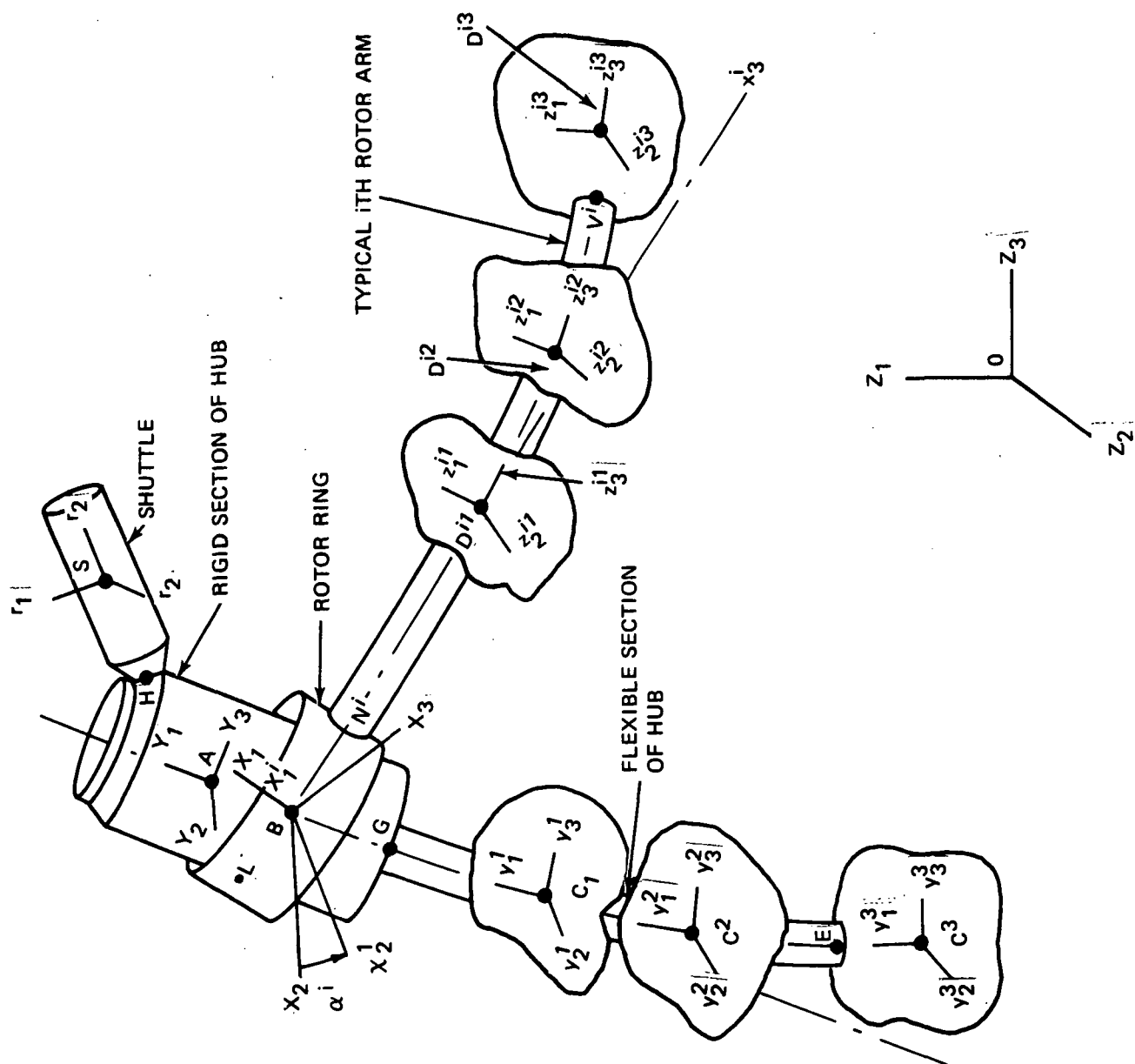


Figure 6-1. Vehicle Idealization

- The flexible section of hub consists of three masses ($p = 3$), the inner two of which are mass points having rotary inertia. The five reference points in the hub, namely the four mass centers and the bearing connection point, are aligned in the undeformed vehicle.
- The two types of symmetric rotors studied are designated as T and Y. The T rotor has two identical arms ($q=2$) 180° apart, and the Y rotor has three identical arms ($q=3$) 120° apart. In both cases, a rotor arm consists of three masses ($s=3$), the inner two of which are mass points having rotary inertia. All mass centers, including that of the rotor ring, lie in a plane; also, the rotor ring mass center coincides with the bearing connection point.
- A docked shuttle may or may not be present; the docking mechanism exhibits elastic and viscous effects (see Equation 6-37)
- The flexible bearing between the hub and the rotor ring exhibits elastic and viscous effects; associated stiffness and damping expressions are given by Equations 6-32 to 6-36. This bearing is discussed in detail in Section 6.3
- The massless structures of the flexible hub and the rotor arms consist of straight beam segments of circular cross-section; the associated stiffness and damping matrices for these structures, discussed in Section 6-2, are integrated into the equations of motion by means of Equations 6-38 to 6-42.

Equations 6-43 contains 84 equations in 84 variables, all of which are needed to describe the motion of the largest simulation vehicle, viz., the vehicle with the Y rotor and the docked shuttle. For computer programming purposes, however, it was decided to preserve the size of the problem when a smaller vehicle was to be studied. This was accomplished by incorporating scalar parameters n_S and n_R into those equations of motion associated with the shuttle and with masses in the third arm of the rotor. For example, when the shuttle is not present ($m_S = 0$, $[I_H]=0$) n_S is set to zero and Euler's equations for the shuttle reduce to $\bar{I} \{\ddot{\Psi}\}=0^1$; then starting from initial conditions $\{\Psi\} = \{\dot{\Psi}\}=0$, the components of $\{\Psi\}$ become meaningless variables with no effect in the analysis. When the shuttle is present, n_S is set to unity and Euler's equations for the shuttle are retained in their usual form. Parameter n_R plays the same role for the third arm of the rotor.

1 - Arbitrary scalar \bar{I} is included in order to avoid introducing ill-conditioning effects into the mass matrix.

Although the equations of 6-43 do constitute a legitimate set of equations of motion, they have certain shortcomings for the simulation study. First, they are difficult to use if the motion is constrained in some way, as in the case of a rigid rotor, etc. Second, when used in conjunction with a modal reduction scheme they do not lead to a particularly good reduced set of equations. Both of these difficulties stem from the fact that the forces on the right-hand side of 6-43 do not constitute a set of generalized forces for the problem. Accordingly, it was decided to premultiply both sides of Equation 6-43 by a transformation matrix such that the resulting right-hand side would contain a set of generalized forces¹. This matrix, denoted as $[Q_{GEN}]$, is presented in Equation (6-44); the associated expressions needed to form $[Q_{GEN}]$ are given by Equation (6-45).

The large size of the problem (84 degrees of freedom) plus the fact that the "λ" and "Q" matrices vary with time made it impractical to use the entire set of equations for the time history study. Accordingly, the modal reduction technique was used to obtain a reduced set of approximate equations in terms of modal coordinates; the degree of approximation depends, of course, on the number of modes used. The essential relationships are presented in Equations (6-46) through (6-49). Equation (6-49) contains two alternate expressions for determining initial values of the modal coordinates. The simpler of the two, (6-49a), was adequate for the needs of the simulation study and was therefore incorporated into the computer program. However, if more complicated initial conditions are to be studied in the future, it would be advisable to use (6-49b).

1 - Specifically, the resulting set of generalized forces are associated with the following coordinates: $\{Z_A\}$, $\{\gamma\}$, $\{\psi\}$, $\{u_C1\}$, $\{\phi_C1\}$, ... $\{u_C3\}$, $\{\phi_C3\}$, $\{\theta\}$, $\{u_D11\}$, $\{\phi_D11\}$, ... $\{u_D33\}$, $\{\phi_D33\}$.

TABLE 6-1

NOTATIONPoints

A	Mass center of rigid section of hub
B	Fixed point in hub about which rotor rotates
C^j	Mass center of j^{th} mass module in flexible section of hub
D^{ik}	Mass center of k^{th} module in i^{th} rotor arm
E	Connection between flexible section of hub and end module of hub
G	Connection between flexible section of hub and right section of hub
H	Connection between hub and shuttle
L	Mass center of rotor ring
N^i	Intersection of rotor ring and i^{th} rotor arm
O	Origin of inertial frame
S	Mass center of shuttle
V^i	Connection between i^{th} rotor arm and associated end module

Coordinate Frames

Z	Inertial frame fixed at O
\tilde{Y}	Frame fixed in rigid section of hub at A; the Y_1 axis is parallel to the nominal spin axis of hub
X	Frame fixed in rotor ring at B; the X_1 axis is nominally aligned with Y_1 .
\tilde{x}^i	Frame fixed in rotor ring at B; x_1^i coincides with X_1 and z_3^i is nominally aligned with i^{th} rotor arm
\tilde{y}^j	Frame fixed in j^{th} mass module of flexible hub at C^j ; this frame is nominally aligned with \tilde{Y}
\tilde{z}^{ik}	Frame fixed in k^{th} mass module of i^{th} rotor arm at D^{ik} , this frame is nominally aligned with \tilde{x}^i
\tilde{x}	Frame fixed in shuttle at S.

Displacements

$\{\gamma\}$	Ordered rotation which comprise transformation from \underline{Z} to \underline{Y} ; i) γ_1 about Z_1 , ii) γ_2 about "carried" Z_2 , iii) γ_3 about "twice carried" Z_3
$\{\theta\}$	Ordered rotations comprising transformation from \underline{Y} to \underline{X}
α^i	Constant rotation about X_1 transforming from \underline{X} to \underline{z}^i
$\{\phi_{C^j}\}$	small elastic rotations comprising transformation from \underline{Y} to \underline{y}^j
$\{\phi_{D^{ik}}\}$	small elastic rotations comprising transformation from \underline{x}^i to \underline{z}^{ik}
$\{\beta\}$	Constant ordered rotations which comprise transformation from \underline{Y} to nominal position of \underline{r}
$\{\psi\}$	Small elastic rotations comprising transformation from nominal position of \underline{r} to instantaneous position of \underline{r}
$\{u_{C^j}\}$	Small elastic displacements of C^j measured in the \underline{Y} frame
$\{u_{D^{ik}}\}$	Small elastic displacements of D^{ik} measured in the x^i frame

Angular Velocities

$\{\omega^Y\}$	Angular velocity of \underline{Y} frame projected onto itself
$\{\omega^X\}$	Angular velocity of \underline{X} frame projected onto itself
$\{\omega^{x^i}\}$	Angular velocity of \underline{x}^i frame projected onto itself
$\{\omega^{y^j}\}$	Angular velocity of \underline{y}^j frame projected onto itself
$\{\omega^{z^{ik}}\}$	Angular velocity of \underline{z}^{ik} frame projected onto itself
$\{\omega^r\}$	Angular velocity of \underline{r} frame projected onto itself

Transformation Matrices

$[a(\gamma)], [a(\theta)], \dots$	transformation matrices associated with $\gamma_1, \theta_1, \dots$
$[b(\gamma)], [b(\theta)], \dots$	transformation matrices associated with $\gamma_2, \theta_2, \dots$
$[c(\gamma)], [c(\theta)], \dots$	transformation matrices associated with $\gamma_3, \theta_3, \dots$
$[\pi(\gamma)]$	transformation matrix going from \underline{Z} to \underline{Y} ; $[\pi(\gamma)] = [c(\gamma)] [b(\gamma)] [a(\gamma)]$

- $[\pi(\theta)]$ transformation matrix going from \underline{Y} to \underline{X}
 $[\pi(\phi_{C^j})]$ transformation matrix going from \underline{Y} to $\underline{y^j}$
 $[\pi(\phi_{D^{ik}})]$ transformation matrix going from $\underline{x^i}$ to $\underline{z^{ik}}$
 $[\pi(\beta)]$ transformation matrix going from \underline{Y} to nominal position of \underline{r}
 $[\pi(\psi)]$ transformation matrix going from nominal position of \underline{r} to final \underline{r}
 $[a(\alpha^i)]$ transformation matrix going from \underline{x} to $\underline{x^i}$

Physical Properties

- m_A mass of rigid hub section
 m_{C^j} mass of j th module in flexible part of hub
 m_L mass of rotor ring
 $m_{D^{ik}}$ mass of k th module in i th rotor arm
 m_S mass of shuttle
 $[I_A]$ inertia matrix of m_A with respect to A (in \underline{Y} frame)
 $[I_{C^j}]$ inertia matrix of m_{C^j} with respect to C^j (in $\underline{y^j}$ frame)
 $[I_B]$ inertia matrix of m_L with respect to B (in \underline{X} frame)
 $[I_{D^{ik}}]$ inertia matrix of $m_{D^{ik}}$ with respect to D^{ik} (in $\underline{z^{ik}}$ frame)
 $[I_H]$ inertia matrix of m_S with respect to H (in \underline{r} frame)

Forces and Couples

(a) Structural

- $\{-F_{C^j}\}$ structural force acting on m_{C^j} (in \underline{Y} frame)
 $\{-F_{D^{ik}}\}$ structural force acting on $m_{D^{ik}}$ (in $\underline{x^i}$ frame)
 $\{-F_G\}$ structural force exerted by flexible hub on m_A at point G (in \underline{Y} frame)
 $\{-M_{C^j}\}$ structural moment acting on m_{C^j} about C^j (in \underline{Y} frame)
 $\{-M_{D^{ik}}\}$ structural moment acting on $m_{D^{ik}}$ about D^{ik} (in $\underline{x^i}$ frame)
 $\{-M_B\}$ structural moment (exerted by all rotor arms) acting on m_L
 about point B (in \underline{X} frame)
 $\{-M_G\}$ structural moment (exerted by flexible hub) acting on m_A
 about point G (in \underline{Y} frame)

(b) Controls

- $\{\tau_{c j}\}$ control moment applied to $m_{c j}$ (in \tilde{y}^j frame)
- $\{\tau_{D ik}\}$ control moment applied to $m_{D ik}$ (in \tilde{z}^{ik} frame)
- $\{\tau_S\}$ control moment applied to m_S (in \tilde{r} frame)
- $\{\tau_A\}$ control moment applied to m_A (in \tilde{Y} frame)
- $\{\tau_L\}$ control moment applied to m_L (in \tilde{X} frame)

(c) Bearing and Docking

- $\{M_{BEAR}\}$ total moment exerted by bearing on rotor ring (in \tilde{X} frame)
- $\{M_{BS}\}$ elastic part of $\{M_{BEAR}\}$
- $\{M_{BV}\}$ viscous part of $\{M_{BEAR}\}$
- $\{M_{DOCK}\}$ moment exerted by docking mechanism on m_S about point H (in \tilde{r} frame)

Stiffnesses, Viscosities, etc.

- K_{BS1}, K_{BS2} linear stiffness coefficients of bearing
- $[K_{BV}]$ bearing damping matrix
- $[K_{BVH}]$ bearing damping matrix when all damping material is fixed to hub
- $[K_{BVR}]$ bearing damping matrix when all damping material is fixed to rotor
- $[K_{DOCK}]$ stiffness matrix of shuttle docking mechanism
- $[C_{DOCK}]$ damping matrix of shuttle docking mechanism
- $[KTA], [KYA]$ stiffness matrices of "T" and "Y" rotors, respectively
- $[CTA], [CYA]$ damping matrices of "T" and "Y" rotors, respectively
- $[K6HG], [K7H]$ hub stiffness matrices
- $[CHG], [CH]$ hub damping matrices

Miscellaneous

- $\{X_B\}, \{Y_B\},$ etc. coordinates of any point (here taken to be B) in the $X, Y,$ etc. coordinate frames

$[\Gamma(\lambda)]$ cross-product matrix, formed from the elements of one vector, say λ , involved in the cross-product, e.g., if

$$\lambda = \begin{pmatrix} \lambda_1 \\ \lambda_2 \\ \lambda_3 \end{pmatrix} \quad \text{then} \quad [\Gamma(\lambda)] = \begin{bmatrix} 0 & -\lambda_3 & \lambda_2 \\ \lambda_3 & 0 & -\lambda_1 \\ -\lambda_2 & \lambda_1 & 0 \end{bmatrix}$$

$[S(\dot{\gamma})]$, $[S(\dot{\theta})]$ matrices used in conversion between angular velocities $\{\omega^Y\}$, $\{\omega^X\}$ and rates of Euler angles $\{\dot{\gamma}\}$, $\{\dot{\theta}\}$

Ω nominal spin speed of rotor

δ instantaneous spin speed of rotor minus nominal spin speed of rotor

$\{\nabla\}$ similar to $\{\theta\}$ but with δ replacing θ_1

δ_B angular misalignment between Y_1 and X_1 axes

δ_B^* value of δ_B below which K_{BS1} applies and above which K_{BS2} applies

η scalar constant between zero and one;

$\eta = 1$ if all bearing damping material is attached to hub

$\eta = 0$ if all bearing damping material is attached to rotor ring

n_R $n_R = 1$ for a three arm rotor; $n_R = 0$ for a two arm rotor

n_S $n_S = 1$ when a docked shuttle is present; $n_S = 0$ when no shuttle is present

\bar{m} dummy mass value used when $n_R = 0$

\bar{I} dummy inertia value used when $n_R = 0$ or $n_S = 0$

$\{\bar{\lambda}\}$, $[M]$, $[N(\theta)]$
 $\{\mu\}$, $[J]$, $\{\xi\}$, $\{\Delta\}$
 $\{\sigma\}$, $\{\rho\}$ } miscellaneous combinations of terms appearing in the analysis

$[\lambda]$, $\{\sigma\}$ matrices by means of which the equations of motion are presented

$[Q_{GEN}]$ matrix which premultiplies the equations of motion so as to eliminate forces of constraint

$[A^*]$, $[B^*]$, ... $[J^*]$ matrices needed to construct $[Q_{GEN}]$

I. Newton's equations for the entire vehicle; all vectors projected onto \tilde{Y} axes

$$[\lambda_1] \{ \ddot{z}_A \} + [\lambda_2] \{ \dot{\omega}^Y \} + \sum_{j=1}^8 [\lambda_3^j] \{ \ddot{u}_{cj} \} + [\lambda_4] \{ \ddot{\psi} \} + \sum_{k=1}^6 [\lambda_5^{ik}] \{ \ddot{u}_{Dk} \} + [\lambda_6] \{ \ddot{\psi} \} = \{ \sigma_1 \} \quad (6-1)$$

II. Newton's equations for the jth mass of the flexible hub; all vectors projected onto \tilde{Y} axes

$$[\lambda_7^j] \{ \ddot{z}_A \} + [\lambda_8^j] \{ \dot{\omega}^Y \} + [\lambda_9^j] \{ \ddot{u}_{cj} \} = \{ \sigma_2^j \} \quad (6-2)$$

III. Newton's equations for the kth mass of the ith rotor arm; all vectors projected onto \tilde{x}^i axes

$$[\lambda_{10}^{ik}] \{ \ddot{z}_A \} + [\lambda_{11}^{ik}] \{ \dot{\omega}^Y \} + [\lambda_{12}^{ik}] \{ \ddot{u}_{Dk} \} + [\lambda_{13}^{ik}] \{ \ddot{\psi} \} = \{ \sigma_3^{ik} \} \quad (6-3)$$

IV. Euler's equations for the jth mass of the flexible hub; torques taken about point C^j ; all vectors projected onto \tilde{y}^i axes

$$[\lambda_{14}^j] \{ \dot{\omega}^Y \} + [\lambda_{15}^j] \{ \dot{\phi}_{cj} \} = \{ \sigma_4^j \} \quad (6-4)$$

V. Euler's equations for the kth mass of the ith rotor arm; torques taken about point D^{ik} ; all vectors projected onto \tilde{z}^{ik} axes

$$[\lambda_{27}^{ik}] \{ \dot{\omega}^Y \} + [\lambda_{16}^{ik}] \{ \ddot{\psi} \} + [\lambda_{17}^{ik}] \{ \ddot{\theta}_{Dk} \} = \{ \sigma_5^{ik} \} \quad (6-5)$$

VI. Euler's equations for the shuttle; torques taken about point H; all vectors projected onto \tilde{r} axes

$$[\lambda_{18}] \{ \ddot{z}_A \} + [\lambda_{19}] \{ \dot{\omega}^Y \} + [\lambda_{20}] \{ \dot{\psi} \} = \{ \sigma_6 \} \quad (6-6)$$

VII. Euler's equations for the rotor ring; torques taken about point B; all vectors projected onto \tilde{X} axes

$$[\lambda_{21}] \{ \ddot{z}_A \} + [\lambda_{22}] \{ \dot{\omega}^Y \} + [\lambda_{23}] \{ \dot{\psi} \} = \{ \sigma_7 \} \quad (6-7)$$

VIII. Euler's equations for the rigid hub/shuttle combination; torques taken about point B; all vectors projected onto \tilde{Y} axes

$$[\lambda_{24}] \{ \ddot{z}_A \} + [\lambda_{25}] \{ \dot{\omega}^Y \} + [\lambda_{26}] \{ \dot{\psi} \} = \{ \sigma_8 \} \quad (6-8)$$

Essential Equations and Definitions

The following set of relationships are required for the calculation of the " λ " and " σ " matrices which constitute the fundamental matrix equations of motion, 6-1 through 6-8.

Coordinate Transformation Matrices

$$[a(\gamma)] \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \gamma_1 & \sin \gamma_1 \\ 0 & -\sin \gamma_1 & \cos \gamma_1 \end{bmatrix} \quad (6-9a)$$

$$[b(\gamma)] \begin{bmatrix} \cos \gamma_2 & 0 & -\sin \gamma_2 \\ 0 & 1 & 0 \\ \sin \gamma_2 & 0 & \cos \gamma_2 \end{bmatrix} \quad (6-9b)$$

$$[c(\gamma)] \begin{bmatrix} \cos \gamma_3 & \sin \gamma_3 & 0 \\ -\sin \gamma_3 & \cos \gamma_3 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (6-9c)$$

$$[\pi(\gamma)] = [c(\gamma)] [b(\gamma)] [a(\gamma)] \quad (6-10)$$

Relationships between angular velocities and time rates of Euler angles

$$\{\dot{\delta}\} = [S(\delta)]^{-1} \{\omega^y\} \quad (6-11)$$

$$\{\dot{\theta}\} = [S(\theta)]^{-1} (\{\omega^x\} - [\pi(\delta)] \{\omega^y\}) \quad (6-12)$$

$$\{\omega^{x^i}\} = [a(\alpha^i)] \{\omega^x\} \quad (6-13)$$

$$\{\omega^r\} = [J] \{\omega^y\} + \{\dot{\psi}\} \quad (6-14)$$

$$\{\omega^{y^j}\} = [\pi(\phi_{c^j})] \{\omega^y\} + \{\dot{\phi}_{c^j}\} \quad (6-15)$$

$$\{\omega^{z^k}\} = [\pi(\phi_{D^k})] [a(\alpha^i)] \{\omega^x\} + \{\dot{\phi}_{D^k}\} \quad (6-16)$$

$$S(\theta) \begin{bmatrix} \cos \theta_3 \cos \theta_2 & \sin \theta_3 & 0 \\ -\sin \theta_3 \cos \theta_2 & \cos \theta_3 & 0 \\ \sin \theta_2 & 0 & 1 \end{bmatrix} \quad (6-17)$$

$$[S(\theta)]^{-1} = \frac{1}{\cos \theta_2} \begin{bmatrix} \cos \theta_3 & -\sin \theta_3 & 0 \\ \sin \theta_3 \cos \theta_2 & \cos \theta_3 \cos \theta_2 & 0 \\ -\cos \theta_3 \sin \theta_2 & \sin \theta_3 \sin \theta_2 & \cos \theta_2 \end{bmatrix} \quad (6-18)$$

Miscellaneous Expressions

$$\{\nabla\} = \begin{Bmatrix} \delta \\ \theta_2 \\ \theta_3 \end{Bmatrix} \quad (6-19)$$

$$\{\dot{\theta}\} = \{\dot{\nabla}\} + \Omega \begin{Bmatrix} 1 \\ 0 \\ 0 \end{Bmatrix} - [\mu] [\pi(\theta)] \{\omega^y\} \quad (6-20)$$

$$[J] = [\pi(\psi)] [\pi(\beta)] \quad (6-21)$$

$$[N(\theta)] = [1] - [s(\theta)] [\mu] \quad (6-22)$$

$$[\mu] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad (6-23)$$

$$\{p\} = [N(\theta)] [\dot{\pi}(\theta)] \{\omega^y\} + [s(\theta)] \{\dot{\theta}\} \quad (6-24)$$

$$[\dot{\pi}(\theta)] = [\Gamma] [s(\theta)] \{\dot{\theta}\} [\pi(\theta)] \quad (6-25)$$

$$[s(\theta)] = \begin{bmatrix} 0 & \dot{\theta}_3 & 0 \\ -\dot{\theta}_3 & 0 & 0 \\ \dot{\theta}_2 & 0 & 0 \end{bmatrix} \quad (6-26)$$

$$[M_1] = -[\pi(\theta)]^T [\Gamma(\chi_L)]$$

$$[M_2^{ik}] = ([a(\alpha^i)] [\pi(\theta)])^T$$

$$[M_3^{ik}] = -[M_2^{ik}] [\Gamma(\chi_{0^{ik}} + \mu_{0^{ik}})] [a(\alpha^i)]$$

$$[M_4] = [J]^T [\Gamma(r_H)] [J] - [\Gamma(\gamma_H)]$$

$$[M_5] = [J]^T [\Gamma(r_H)]$$

(6-27)

$$\{\xi_1^j\} = 2[\Gamma(\omega^y)] \{\dot{\mu}_{c^j}\} + [\Gamma(\omega^y)]^2 \{\gamma_{c^j} + \mu_{c^j}\}$$

$$\{\xi_2\} = [\Gamma(\omega^y)]^2 \{\gamma_B\} + [\pi(\theta)]^T [\Gamma(\omega^x)]^2 \{\chi_L\}$$

$$\{\xi_3^{ik}\} = [\Gamma(\omega^y)]^2 \{\gamma_B\} + [M_2^{ik}] (2[\Gamma(\omega^{x^i})] \{\dot{\mu}_{0^{ik}}\} + [\Gamma(\omega^{x^i})]^2 \{\chi_{0^{ik}} + \mu_{0^{ik}}\})$$

$$\{\xi_4\} = [\Gamma(\omega^y)]^2 \{\gamma_H\} - [J]^T ([\Gamma(\omega^r)]^2 \{\gamma_H\} + [\Gamma(r_H)] [\Gamma(\psi)] [\pi(\beta)] \{\omega^y\})$$

(6-28)

$$\{\Delta_1\} = \sum_{j=1}^0 (m_{c^j} \{\xi_1^j\}) + m_L \{\xi_2\} + \sum_{i=1}^0 \sum_{k=1}^0 (m_{0^{ik}} \{\xi_3^{ik}\}) + m_s \{\xi_4\}$$

$$\{\Delta_2^j\} = m_{c^j} \{\xi_1^j\}$$

$$\{\Delta_3^{ik}\} = m_{0^{ik}} [a(\alpha^i)] [\pi(\theta)] \{\xi_3^{ik}\}$$

$$\{\Delta_4^j\} = -[I_{c^j}] [\Gamma(\psi_{c^j})] \{\omega^y\} + [\Gamma(\omega^{y^j})] [I_{c^j}] \{\omega^{y^j}\}$$

$$\{\Delta_5^{ik}\} = -[I_{0^{ik}}] [\Gamma(\psi_{0^{ik}})] [a(\alpha^i)] \{\omega^x\} + [\Gamma(\omega^{3^{ik}})] [I_{0^{ik}}] \{\omega^{3^{ik}}\}$$

$$\{\Delta_6\} = -m_s [\Gamma(r_H)] [J] [\Gamma(\omega^y)]^2 \{\gamma_H\} - [I_H] [\Gamma(\psi)] [\pi(\beta)] \{\omega^y\} + [\Gamma(\omega^r)] [I_H] \{\omega^r\}$$

$$\{\Delta_7\} = m_L [\Gamma(\chi_L)] [\pi(\theta)] [\Gamma(\omega^y)]^2 \{\gamma_B\} + [\Gamma(\omega^x)] [I_B] \{\omega^x\}$$

$$\{\Delta_8\} = [\Gamma(\omega^y)] [I_A] \{\omega^y\} + [J]^T ([\Gamma(\omega^r)] [I_H] \{\omega^r\} - [I_H] [\Gamma(\psi)] [\pi(\beta)] \{\omega^y\})$$

$$+ m_s [\Gamma(\gamma_H - \gamma_B)] \{\xi_4\} - m_s [J]^T [\Gamma(r_H)] [J] [\Gamma(\omega^y)]^2 \{\gamma_H\}$$

(6-29)

Tabulated "σ" Vectors

$$\{\bar{\sigma}_1\} = -\{\Delta_1\}$$

$$\{\sigma_1\} = \{\bar{\sigma}_1\} - [\bar{\lambda}_4] \{\rho\}$$

$$\{\sigma_2^j\} = -\{\Delta_2^j\} - \{E_{c^j}\}$$

$$\{\bar{\sigma}_3^{ik}\} = -\{\Delta_3^{ik}\} - \{F_{D^{ik}}\}$$

$$\{\sigma_3^{ik}\} = \{\bar{\sigma}_3^{ik}\} - [\bar{\lambda}_{13}^{ik}] \{\rho\}$$

$$\{\sigma_4^j\} = -\{\Delta_4^j\} - [\pi(\phi_{c^j})] \{M_{c^j}\} + \{\tau_{c^j}\}$$

$$\{\bar{\sigma}_5^{ik}\} = -\{\Delta_5^{ik}\} - [\pi(\phi_{D^{ik}})] \{M_{D^{ik}}\} + \{\tau_{D^{ik}}\}$$

$$\{\sigma_5^{ik}\} = \{\bar{\sigma}_5^{ik}\} - [\bar{\lambda}_{16}^{ik}] \{\rho\}$$

$$\{\sigma_6\} = -\{\Delta_6\} + \{\tau_s\} + \{M_{DOCK}\}$$

(6-30)

$$\{\bar{\sigma}_7\} = -\{\Delta_7\} - \{M_B\} + \{\tau_L\} + \{M_{BEAR}\}$$

$$\{\sigma_7\} = \{\bar{\sigma}_7\} - [\bar{\lambda}_{23}] \{\rho\}$$

$$\{\sigma_8\} = -\{\Delta_8\} + \{\tau_A\} + [J]^T \{\tau_s\} - [\pi(\Theta)]^T \{M_{BEAR}\} - \{M_G\} - [\Gamma(Y_G - Y_B)] \{F_G\}$$

Tabulated "λ" Matrices

$$[\lambda_1] = (m_A + \sum_{j=1}^B m_{c_j} + m_L + \sum_{i=1}^B \sum_{k=1}^A m_{D^{ik}} + m_S) [\pi(\gamma)]$$

$$[\bar{\lambda}_2] = m_S [M_4] - (m_L + \sum_{i=1}^B \sum_{k=1}^A m_{D^{ik}}) [\Gamma(\gamma_B)] - \sum_{j=1}^B (m_{c_j} [\Gamma(\gamma_{c_j} + \mu_{c_j})])$$

$$[\lambda_2] = [\bar{\lambda}_2] + [\bar{\lambda}_4] [N(\theta)] [\pi(\theta)]$$

$$[\lambda_3^j] = m_{c_j} [1]$$

$$[\bar{\lambda}_4] = m_L [M_1] + \sum_{i=1}^B \sum_{k=1}^A (m_{D^{ik}} [M_3^{ik}])$$

$$[\lambda_4] = [\bar{\lambda}_4] [s(\theta)]$$

$$[\lambda_5^{ik}] = m_{D^{ik}} [M_2^{ik}]$$

$$[\lambda_6] = m_S [M_5]$$

$$[\lambda_7^j] = m_{c_j} [\pi(\gamma)]$$

$$[\lambda_8^j] = -m_{c_j} [\Gamma(\gamma_{c_j} + \mu_{c_j})]$$

$$[\lambda_9^j] = m_{c_j} [1] = [\lambda_3^j]$$

$$[\lambda_{10}^{ik}] = m_{D^{ik}} [a(\alpha^i)] [\pi(\theta)] [\pi(\gamma)]$$

$$[\bar{\lambda}_{11}^{ik}] = -m_{D^{ik}} [a(\alpha^i)] [\pi(\theta)] [\Gamma(\gamma_B)]$$

$$[\lambda_{11}^{ik}] = [\bar{\lambda}_{11}^{ik}] + [\bar{\lambda}_{13}^{ik}] [N(\theta)] [\pi(\theta)]$$

$$[\lambda_{12}^{ik}] = m_{D^{ik}} [1]$$

$$[\lambda_{12}^{*3k}] = \eta_R [\lambda_{12}^{3k}] + \bar{m} (1 - \eta_R) [1]$$

$$[\bar{\lambda}_{13}^{ik}] = -m_{D^{ik}} [\Gamma(\chi_{D^{ik}}^i + \mu_{D^{ik}})] [a(\alpha^i)]$$

$$[\lambda_{13}^{ik}] = [\bar{\lambda}_{13}^{ik}] [s(\theta)]$$

$$[\lambda_{14}^j] = [I_{c_j}] [\pi(\phi_{c_j})]$$

$$[\lambda_{15}^j] = [T_i]$$

(6-31)

$$[\bar{\lambda}_{16}^{ik}] = [I_{D^{ik}}][\pi(\phi_{D^{ik}})][a(\alpha^i)]$$

$$[\lambda_{16}^{ik}] = [\bar{\lambda}_{16}^{ik}][s(\theta)]$$

$$[\lambda_{17}^{ik}] = [I_{D^{ik}}]$$

$$[\lambda_{17}^{*3k}] = \pi_R[\lambda_{17}^{*3k}] + \bar{I}(1-\pi_R)[1]$$

$$[\lambda_{18}] = -m_S[\Gamma(r_H)][J][\pi(\gamma)]$$

$$[\lambda_{19}] = m_S[\Gamma(r_H)][J][\Gamma(\gamma_H)] + [I_H][J]$$

$$[\lambda_{20}] = [I_H]$$

$$[\lambda_{20}^{*s}] = \pi_S[\lambda_{20}] + \bar{I}(1-\pi_S)[1]$$

$$[\lambda_{21}] = m_L[\Gamma(\chi_L)][\pi(\theta)][\pi(\gamma)]$$

$$[\bar{\lambda}_{22}] = -m_L[\Gamma(\chi_L)][\pi(\theta)][\Gamma(\gamma_B)]$$

$$[\lambda_{22}] = [\bar{\lambda}_{22}] + [\bar{\lambda}_{23}][N(\theta)][\pi(\theta)]$$

$$[\bar{\lambda}_{23}] = [I_B]$$

$$[\lambda_{23}] = [\bar{\lambda}_{23}][s(\theta)]$$

$$[\lambda_{24}] = (-m_A[\Gamma(\gamma_\theta)] + m_S[\Gamma(\gamma_H - \gamma_\theta)] - m_S[J]^T[\Gamma(r_H)][J])[\pi(\gamma)]$$

$$[\lambda_{25}] = [I_A] + [J]^T[I_H][J] + m_S([\Gamma(\gamma_H - \gamma_\theta)][M_4] + [J]^T[\Gamma(r_H)][J][\Gamma(\gamma)]$$

$$[\lambda_{26}] = [J]^T[I_H] + m_S[\Gamma(\gamma_H - \gamma_B)][M_5]$$

$$[\lambda_{27}^{ik}] = [\bar{\lambda}_{16}^{ik}][N(\theta)][\pi(\theta)]$$

(6-31 Continued)

Note: $[\lambda_{12}^{*3k}]$, $[\lambda_{17}^{*3k}]$ and $[\lambda_{20}]$ apply to the vehicle simulation study only.

Expressions for Flexible Bearing and Flexible Docking Mechanism

The following equations for the bearing and docking mechanism were used in the vehicle simulation study:

Bearing Damping

$$[K_{BVH}] = [K_{BVR}] = \begin{bmatrix} 0 & 0 & 0 \\ 0 & E_B & 0 \\ 0 & 0 & E_B \end{bmatrix} \quad (6-32a)$$

$$[K_{BV}] = \eta [K_{BVH}] + (1-\eta) [\pi(\theta)] [K_{BVR}] [\pi(\theta)]^T, \quad 0 \leq \eta \leq 1 \quad (6-32b)$$

$$\{M_{BV}\} = -[K_{BV}] (\{w^x\} - [\pi(\theta)] \{w^y\}) \quad (6-33)$$

Bearing Stiffness

$$\{\delta_B\} = \arccos(\cos \theta_2 \cos \theta_3) \quad (6-34)$$

$$\{M_{BS}\} = -K_{BS1} \begin{Bmatrix} 0 \\ \theta_2 \\ \theta_3 \end{Bmatrix} \quad \text{for } \delta_B \leq \delta_B^* \quad (6-35a)$$

$$\{M_{BS}\} = \left((K_{BS2} - K_{BS1}) \frac{\delta_B^*}{\delta_B} - K_{BS2} \right) \begin{Bmatrix} 0 \\ \theta_2 \\ \theta_3 \end{Bmatrix} \quad \text{for } \delta_B > \delta_B^* \quad (6-35b)$$

Note: δ_B^* should not be less than 10^{-6} rad.

Total Bearing Moment

$$\{M_{BEAR}\} = \{M_{BV}\} + \{M_{BS}\} \quad (6-36)$$

Docking Moment

$$\{M_{DOCK}\} = -[K_{DOCK}] \{\psi\} - [C_{DOCK}] \{\dot{\psi}\} \quad (6-37)$$

Structural Forces

The following expressions are for the hub and rotor structures used in the simulation study:

Hub

$$\begin{Bmatrix} F_{G'} \\ F_{HUB} \end{Bmatrix} = \begin{bmatrix} [K6HG] \\ [K7H] \end{bmatrix} \{q_{HUB}\} + \begin{bmatrix} [CHG] \\ [CH] \end{bmatrix} \{\dot{q}_{HUB}\} \quad (6-38)$$

where

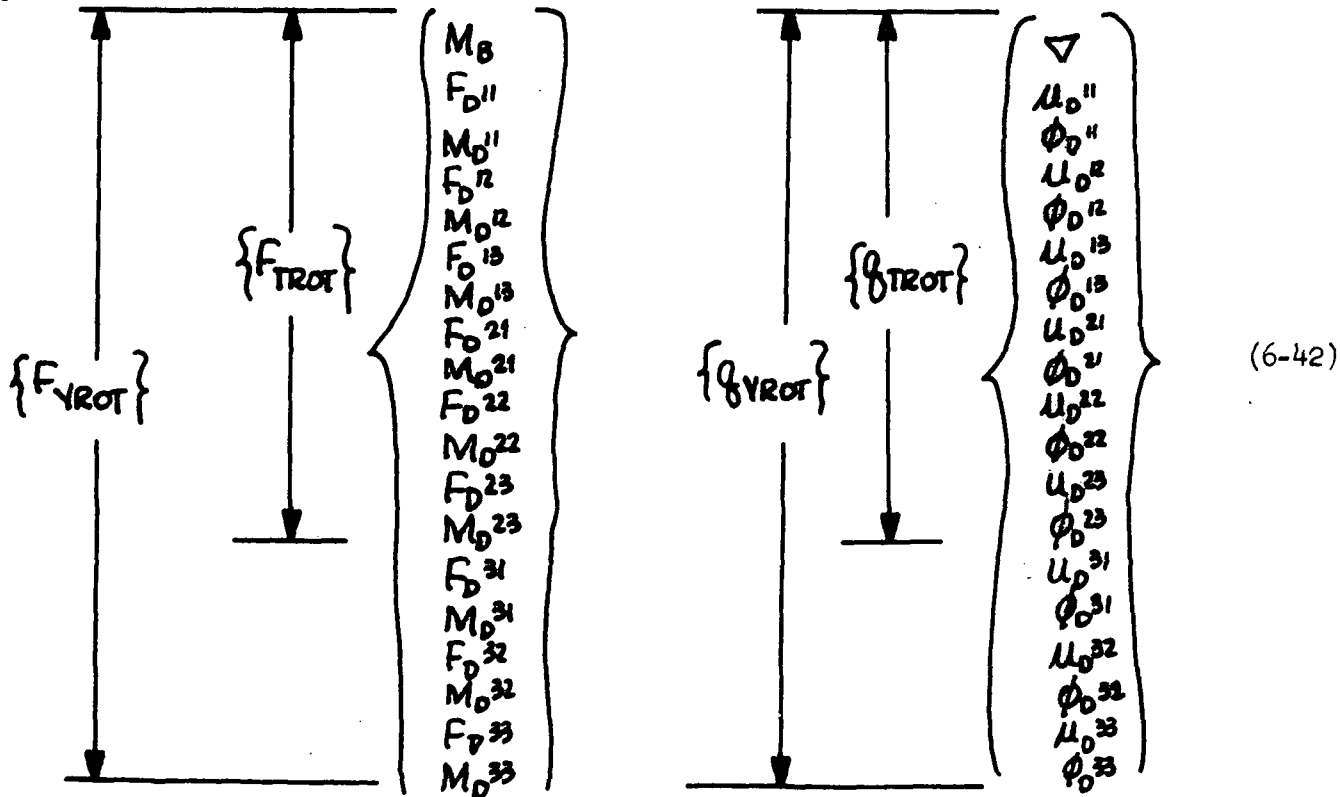
$$\{F_{G'}\} = \begin{Bmatrix} F_G \\ M_G \end{Bmatrix}, \quad \{F_{HUB}\} = \begin{Bmatrix} F_{c1} \\ M_{c1} \\ F_{c2} \\ M_{c2} \\ F_{c3} \\ M_{c3} \end{Bmatrix}, \quad \{q_{HUB}\} = \begin{Bmatrix} u_{c1} \\ \phi_{c1} \\ u_{c2} \\ \phi_{c2} \\ u_{c3} \\ \phi_{c3} \end{Bmatrix} \quad (6-39)$$

Rotor

$$\{F_{TROT}\} = [KTA] \{q_{TROT}\} + [CTA] \{\dot{q}_{TROT}\} \quad (6-40)$$

$$\{F_{YROT}\} = [KYA] \{q_{YROT}\} + [CYA] \{\dot{q}_{YROT}\} \quad (6-41)$$

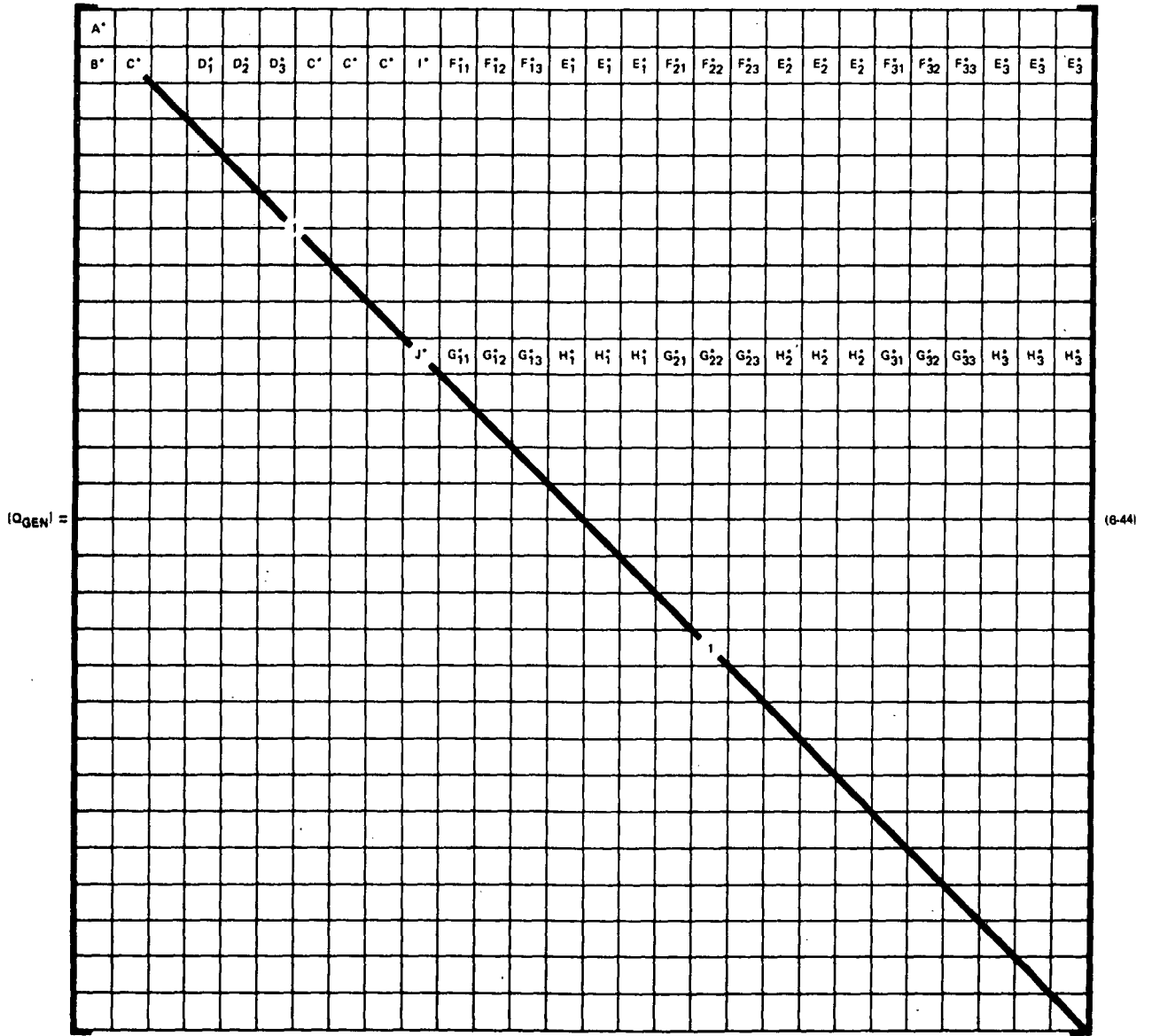
where



(6-43)

NEWTON EQUATION SYSTEM	\dot{p}_1	\dot{p}_2	\dot{p}_3	\dot{p}_4	\dot{p}_5	\dot{p}_6	\dot{p}_7	\dot{p}_8	\dot{p}_9	\dot{p}_{10}	\dot{p}_{11}	\dot{p}_{12}	\dot{p}_{13}	\dot{p}_{14}	\dot{p}_{15}	\dot{p}_{16}	\dot{p}_{17}	\dot{p}_{18}	\dot{p}_{19}	\dot{p}_{20}	\dot{p}_{21}	\dot{p}_{22}	\dot{p}_{23}	\dot{p}_{24}	\dot{p}_{25}	\dot{p}_{26}	\dot{p}_{27}	\dot{p}_{28}	\dot{p}_{29}	\dot{p}_{30}	\dot{p}_{31}	\dot{p}_{32}	\dot{p}_{33}	\dot{p}_{34}	\dot{p}_{35}	\dot{p}_{36}	\dot{p}_{37}	\dot{p}_{38}	\dot{p}_{39}	\dot{p}_{40}	\dot{p}_{41}	\dot{p}_{42}	\dot{p}_{43}	\dot{p}_{44}	\dot{p}_{45}	\dot{p}_{46}	\dot{p}_{47}	\dot{p}_{48}	\dot{p}_{49}	\dot{p}_{50}	\dot{p}_{51}	\dot{p}_{52}	\dot{p}_{53}	\dot{p}_{54}	\dot{p}_{55}	\dot{p}_{56}	\dot{p}_{57}	\dot{p}_{58}	\dot{p}_{59}	\dot{p}_{60}	\dot{p}_{61}	\dot{p}_{62}	\dot{p}_{63}	\dot{p}_{64}	\dot{p}_{65}	\dot{p}_{66}	\dot{p}_{67}	\dot{p}_{68}	\dot{p}_{69}	\dot{p}_{70}	\dot{p}_{71}	\dot{p}_{72}	\dot{p}_{73}	\dot{p}_{74}	\dot{p}_{75}	\dot{p}_{76}	\dot{p}_{77}	\dot{p}_{78}	\dot{p}_{79}	\dot{p}_{80}	\dot{p}_{81}	\dot{p}_{82}	\dot{p}_{83}	\dot{p}_{84}	\dot{p}_{85}	\dot{p}_{86}	\dot{p}_{87}	\dot{p}_{88}	\dot{p}_{89}	\dot{p}_{90}	\dot{p}_{91}	\dot{p}_{92}	\dot{p}_{93}	\dot{p}_{94}	\dot{p}_{95}	\dot{p}_{96}	\dot{p}_{97}	\dot{p}_{98}	\dot{p}_{99}	\dot{p}_{100}
CONSTRAINT EQUATION SYSTEM	\dot{q}_1	\dot{q}_2	\dot{q}_3	\dot{q}_4	\dot{q}_5	\dot{q}_6	\dot{q}_7	\dot{q}_8	\dot{q}_9	\dot{q}_{10}	\dot{q}_{11}	\dot{q}_{12}	\dot{q}_{13}	\dot{q}_{14}	\dot{q}_{15}	\dot{q}_{16}	\dot{q}_{17}	\dot{q}_{18}	\dot{q}_{19}	\dot{q}_{20}	\dot{q}_{21}	\dot{q}_{22}	\dot{q}_{23}	\dot{q}_{24}	\dot{q}_{25}	\dot{q}_{26}	\dot{q}_{27}	\dot{q}_{28}	\dot{q}_{29}	\dot{q}_{30}	\dot{q}_{31}	\dot{q}_{32}	\dot{q}_{33}	\dot{q}_{34}	\dot{q}_{35}	\dot{q}_{36}	\dot{q}_{37}	\dot{q}_{38}	\dot{q}_{39}	\dot{q}_{40}	\dot{q}_{41}	\dot{q}_{42}	\dot{q}_{43}	\dot{q}_{44}	\dot{q}_{45}	\dot{q}_{46}	\dot{q}_{47}	\dot{q}_{48}	\dot{q}_{49}	\dot{q}_{50}	\dot{q}_{51}	\dot{q}_{52}	\dot{q}_{53}	\dot{q}_{54}	\dot{q}_{55}	\dot{q}_{56}	\dot{q}_{57}	\dot{q}_{58}	\dot{q}_{59}	\dot{q}_{60}	\dot{q}_{61}	\dot{q}_{62}	\dot{q}_{63}	\dot{q}_{64}	\dot{q}_{65}	\dot{q}_{66}	\dot{q}_{67}	\dot{q}_{68}	\dot{q}_{69}	\dot{q}_{70}	\dot{q}_{71}	\dot{q}_{72}	\dot{q}_{73}	\dot{q}_{74}	\dot{q}_{75}	\dot{q}_{76}	\dot{q}_{77}	\dot{q}_{78}	\dot{q}_{79}	\dot{q}_{80}	\dot{q}_{81}	\dot{q}_{82}	\dot{q}_{83}	\dot{q}_{84}	\dot{q}_{85}	\dot{q}_{86}	\dot{q}_{87}	\dot{q}_{88}	\dot{q}_{89}	\dot{q}_{90}	\dot{q}_{91}	\dot{q}_{92}	\dot{q}_{93}	\dot{q}_{94}	\dot{q}_{95}	\dot{q}_{96}	\dot{q}_{97}	\dot{q}_{98}	\dot{q}_{99}	\dot{q}_{100}

Matrix Equations of Motion for Simulation Vehicles



(QGEN) MATRIX FOR SIMULATION VEHICLES

Submatrices of $[Q_{GEN}]$

$$[A^*] = [\pi(\gamma)]^T$$

$$[B^*] = [s(\gamma)]^T [\Gamma(\gamma_B)]$$

$$[C^*] = [s(\gamma)]^T$$

$$[D_j^*] = [s(\gamma)]^T [\Gamma(\gamma_{c_j} + u_{c_j} - \gamma_B)]$$

$$[E_i^*] = [s(\gamma)]^T [\pi(\theta)]^T [a(\alpha^i)]^T$$

$$[F_{ik}^*] = [E_i^*] [\Gamma([a(\alpha^i)] [\pi(\theta)] \{ \gamma_B \} + \{ \chi_{p_{ik}}^i + \mu_{p_{ik}} \})] - [B^*] [\pi(\theta)]^T [a(\alpha^i)]^T$$

$$[G_{ik}^*] = [H_i^*] [\Gamma(\chi_{p_{ik}}^i + \mu_{p_{ik}})]$$

$$[H_i^*] = [s(\theta)]^T [a(\alpha^i)]^T$$

$$[I^*] = [s(\gamma)]^T [\pi(\theta)]^T$$

$$[J^*] = [s(\theta)]^T$$

(6-45)

Modal Reduction

Original equations of motion in terms of 84 physical coordinates $\{x\}$:

$$[Q_{GEN}][\lambda] \{ \ddot{x} \} = [Q_{GEN}] \{ \sigma \}$$

(6-46)

Modal approximation; n modes $[\phi]$; n modal coordinates $\{q\}$:

$$\{x\} = [\phi] \{q\} \quad (6-47)$$

Reduced set of approximate equations in modal coordinates:

$$([\phi]^T [Q_{GEN}] [\lambda] [\phi]) \{\ddot{q}\} = [\phi]^T [Q_{GEN}] \{\sigma\} \quad (6-48)$$

Initial conditions in modal coordinates:

$$\{q\} = ([\phi]^T [\phi])^{-1} [\phi]^T \{x\} \quad (6-49a)$$

or

$$\{q\} = ([\phi]^T [Q_{GEN}] [\lambda] [\phi])^{-1} [\phi]^T [Q_{GEN}] [\lambda] \{x\} \quad (6-49b)$$

6.2 Formulation of Structural Characteristics

The details of the procedure by means of which structural modes, frequencies, stiffness matrices and damping matrices were computed are lengthy and are presented in Appendix A. Major features of the formulation are as follows:

- The rigid section of hub was considered to be fixed in space; consequently, cantilever modes were computed for the flexible section of hub.
- Nonrotating modes for the rotors were computed assuming that the rotor ring was pinned to ground (rigid section of hub); rotation of the rotor ring out of the spin plane was resisted by rotary springs having the elastic characteristics of the bearing.
- The stiffening effects due to centrifugal forces in a spinning rotor were accounted for in the analysis by incorporating an incremental beam stiffness matrix; for the modal analysis, the centrifugal forces were represented by a properly oriented set of external loads.

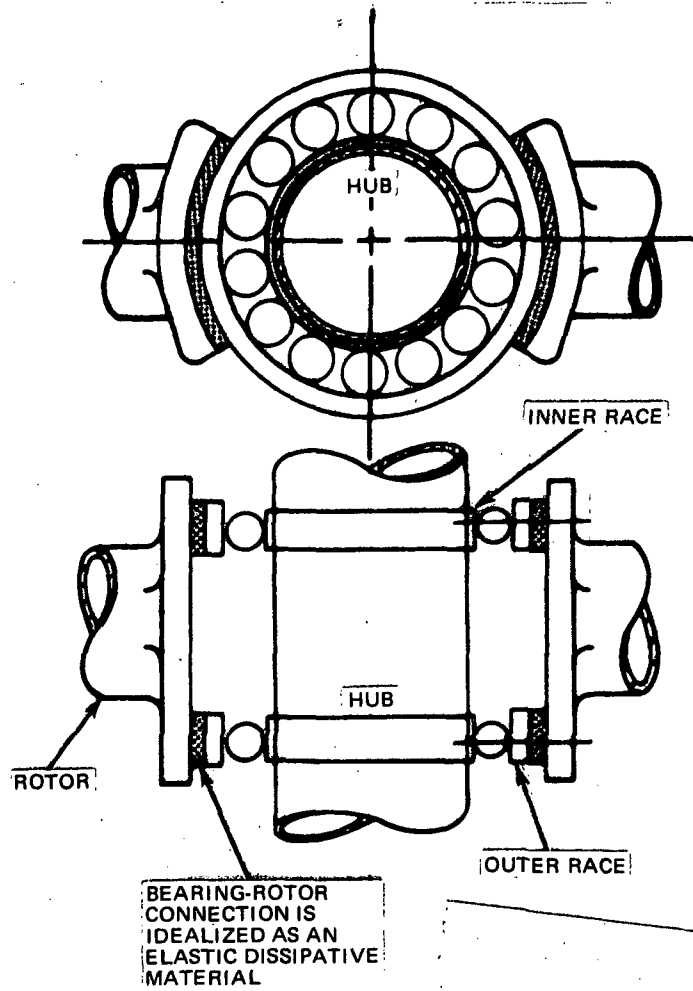
- Due to symmetry of the rotor, some repeated eigenvalues and coupled modes were anticipated. Although mathematically acceptable, this condition could make the mode shapes difficult to interpret. To avoid this problem, rotor motion was divided into its symmetric and antisymmetric parts and two separate eigenvalue problems were solved for each rotor.
- The rotor had one rigid body mode, namely simple rotation in the spin plane. Since the eigenvalue algorithm used could not deal with a zero frequency, it was necessary to eliminate this frequency from the equations. This was accomplished by using a simple frequency shift technique (See Appendix B) rather than the conventional reduction technique.
- With some minor modification, structural damping matrices were based on modal damping.

6.3 Rotor-Hub Bearing Math Model

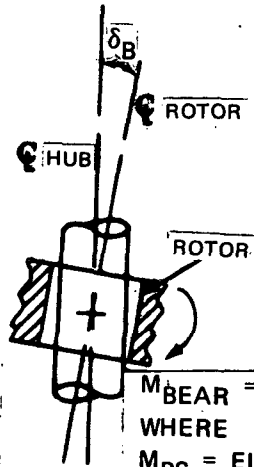
Calculations for the bearing stiffness and damping parameters are based on an assumed model illustrated in Figure 6-2. In order to assure a tight fit and to compensate for wear, it was assumed that the bearing will be assembled by heating the outer race and permitting it to shrink over the inner bearing structure as it cools. In Section 4.3.2 an approximate analysis was conducted to determine reasonable elastic and damping parameters, K_{BS1} and E_B , for the bearing. It may be assumed that stiffness K_{BS1} is suitable for deflections δ_B below some critical value, δ_B^* . For larger δ_B a section of the elastic dissipative material is assumed to be completely compressed causing bottoming of the outer race against the rotor; hence the structure then has the higher stiffness K_{BS2} . These characteristics may be summarized by the following equations:

$$M_{BS} = K_{BS1} \delta_B \quad \text{for } \delta_B \leq \delta_B^*$$

$$M_{BS} = [K_{BS2} \delta_B - (K_{BS2} - K_{BS1}) \delta_B^*] \quad \text{for } \delta_B > \delta_B^*$$



DEFLECTION DIAGRAM



$M_{BEAR} = M_{BS} + M_{BV}$
 WHERE
 M_{BS} = ELASTIC RESTORING TORQUE
 M_{BV} = DAMPING TORQUE

Figure 6-2. Elastic Characteristics of Bearing

By setting $K_{BS1} = 0$, the above equations can also be used to investigate the behavior of a structure with a bearing which is so badly worn that slop occurs.

The damping torque exerted on the rotor by the hub is a function of the spin rates of the viscous materials located in the bearing; also, the damping properties of a bearing with dissipative material which is fixed relative to the hub differ from the damping properties when the dissipative material is fixed relative to the rotor. This point will be explained by using the idealizations illustrated in Figure 6-3.

In Figure 6-3a all of the damping material in the bearing is fixed to the hub. A slip ring is assumed to separate the rotor from the non-rotating structure. The vector $\vec{\omega}^*$ denotes the angular velocity of the rotor relative to the hub. If $\vec{\omega}^*$ were directed along the rotor axis ($\vec{\omega}_{R_{2,3}}^* = 0$), the deformed

shape of the damping material would not change with time; therefore, no damping would occur. For damping to occur, $\vec{\omega}^*$ must have a nonzero projection on the rotor transverse plane ($\vec{\omega}_{R_{2,3}}^* \neq 0$). Neglecting nonlinear effects, the

rate of deformation of the damping material is proportional to $\vec{\omega}_{R_{2,3}}^*$;

therefore the damping torque on the rotor is also assumed to be proportional to $\vec{\omega}_{R_{2,3}}^*$ but opposite in direction since it is a restoring torque. Thus,

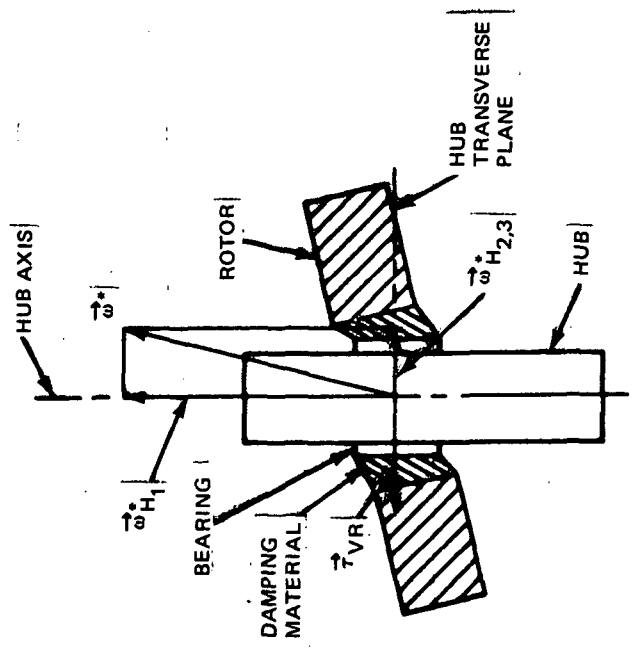
the damping torque on the rotor is given by the following expression when the dissipative material is connected to the hub:

$$\vec{M}_{BVH} = -E_B \vec{\omega}_{R_{2,3}}^*$$

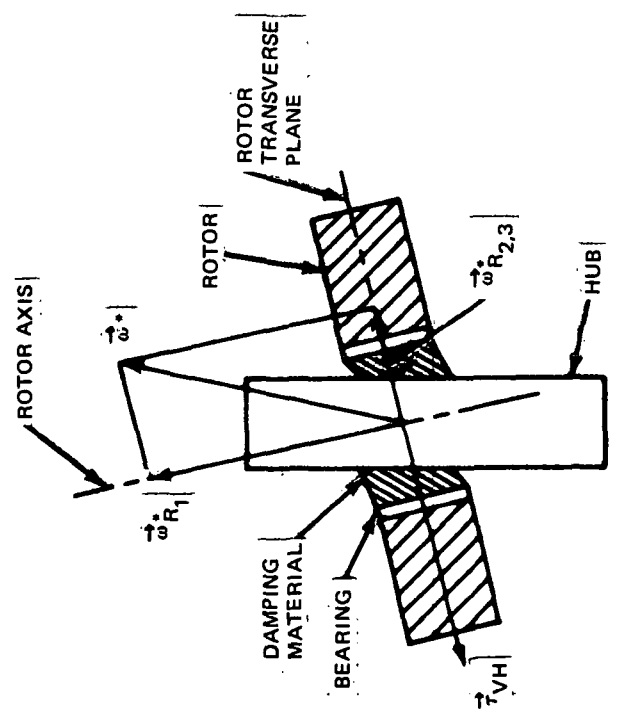
Since E_B is a constant, it may be obtained by testing the nonrotating structure.

Applying the same reasoning the idealization of Figure 6-3b, it can be shown that the damping torque on the rotor would obey the following expression if all of the dissipative material were connected to the rotor:

$$\vec{M}_{BVR} = -E_B \vec{\omega}_{H_{2,3}}^*$$



A. DAMPING MATERIAL CONNECTED TO ROTOR



B. DAMPING MATERIAL CONNECTED TO HUB

Figure 6-3. Idealization Used to Assess Effect of Rotation on Bearing Damping

In practice, it is likely that a combination of the above two effects occurs. This phenomenon was approximated in the equations of motion by assuming that the actual damping torque on the rotor is given by the following weighted combination of the above two effects:

$$\bar{M}_{BV} = \eta \vec{M}_{BVH} + (1-\eta) \vec{M}_{BVR}$$

where η is a scalar parameter between zero and one. If, in particular bearing design, most of the damping material revolves at or near the rotor speed, η would be set close to zero; if the opposite is true, η would be set close to unity. All the essential relations pertaining to the bearing torque are presented in matrix form in equations 6-32 to 6-36.

7.0 Computer Programs

A digital simulation of a flexible, spinning space station, as represented by the equations in Section 6.0, was developed to satisfy the primary objectives of this study. This digital program, referred to as the "General Flexible Body Simulation - Stage II," is described in detail in Part III of this report, where a user's guide is presented. It should be noted that the Stage I simulation, previously developed under contract to MSC (Reference 3), did not include:

- o as many flexible body degrees of freedom
- o a flexible hub
- o a flexible shuttle docking mechanism
- o a capability for modal reduction

Furthermore, the fact that the high frequency modes could not be eliminated from the Stage I simulation made it an inefficient analysis tool. Due to the above noted failings of the Stage I simulation the Stage II simulation was developed. However, results from the Stage I simulation were used as a baseline for comparison during Stage II checkout.

The General Flexible Body - Stage II simulation is divided into two main sub-programs:

1. The TIME HISTORY PROGRAM (THP) - which is a numerical integration of the flexible body equations.
2. The MODAL SERVICE PROGRAM (MSP) - which is used to determine the structural matrices (e.g., stiffness matrix, damping matrix, and modal matrix) that is needed for use by the THP.

Concurrent with the development of the Stage II simulation, preliminary analyses were performed with the previously developed Simplified Flexible Body simulation (Reference 3). This digital program simulates a dual spin space station whose rigid rotor is connected to a rigid hub through a flexible bearing.

8.0 Uncontrolled (Open-Loop) Dynamics

8.1 Preliminaries

Prior to proceeding with a discussion of the complex motion of a flexible rotating space station it is advisable to establish a clear understanding concerning the motion of a rigid spinning body. For demonstration purposes, a simple model for the space station is used; specifically, it is assumed that the spin axis (x axis) is a principle axis for both the hub and the rotor and that the hub is symmetrical. As a result, the Euler equations are identical to those for a spinning body with no hub provided I_y and I_z include both the hub and rotor mass and I_x includes only the rotor mass (see Reference 8). It should be noted that both the "T" and "Y" configurations have a near-symmetric hub. The following inertial angular rates projected on coordinates fixed in the rotating body are obtained by solving the uncontrolled linearized Euler equations:

$$\begin{aligned}\omega_x &= P = \text{constant spin} \\ \omega_y &= F \Omega \alpha \sin(\lambda pt + \beta) \\ \omega_z &= \Omega \cos(\lambda pt + \beta)\end{aligned}\tag{8-1}$$

where

$$\begin{aligned}\lambda &\equiv \sqrt{\lambda_y, \lambda_z}, \quad \alpha \equiv \sqrt{\frac{\lambda_y}{\lambda_z}} \\ \lambda_y &\equiv \frac{I_x - I_z}{I_y}, \quad \lambda_z \equiv \frac{I_x - I_y}{I_z}\end{aligned}\tag{8-2}$$

and Ω and β are determined by the initial conditions. The upper sign in the expression for ω_y applies for "Max I" vehicles ($I_x > I_y, I_x > I_z$), and the lower sign applies for "Min I" vehicles ($I_x < I_y, I_x < I_z$). The case where I_x is an intermediate axis of inertia is not considered since configurations with this property have unstable motions when uncontrolled.

The wobble vector (ω_r) is defined as the projection of the station's angular velocity on the transverse plane; i.e.,

$$\{\omega_T\} = \Omega [\mp \alpha \sin(\lambda pt) \{j\} + \cos(\lambda pt) \{k\}] \quad (8-3)$$

It is noted that the phase angle (β) can be omitted without a significant loss of generality. Examining the above relationship for $\{\omega_T\}$ it is seen that the vector traces an elliptical path as illustrated in Figure 8-1. The frequency of the ellipse, referred to as the wobble frequency (λp), will increase as the separation between the transverse inertias and the spin inertia is increased (see equation 8-2). Furthermore, it is noted that the ellipse degenerates into a circle for symmetric vehicles (i.e., when $\alpha = 1$). Finally, the rotation of the vector is in the same direction as the vehicle spin for a Max I configuration and opposite for Min I.

It is easier to visualize the vehicle motion by observing the wobble rate as seen in despun hub coordinates. The transverse components of the wobble rate as viewed in the hub, is given by

$$\begin{aligned} \omega_{yH} &= (\{\omega_T\} \cdot \{j\}) \cos(pt) - (\{\omega_T\} \cdot \{k\}) \sin(pt) \\ \omega_{zH} &= (\{\omega_T\} \cdot \{j\}) \sin(pt) + (\{\omega_T\} \cdot \{k\}) \cos(pt) \end{aligned} \quad (8-4)$$

Substituting the expression in (8-3) for $\{\omega_T\}$ and rearranging terms results in,

$$\begin{aligned} \omega_{yH} &= \frac{\Omega}{2} \{ (\alpha \mp 1) \cos[p(1-\lambda)t] + (\alpha \pm 1) \cos[p(1+\lambda)t] \} \\ \omega_{zH} &= \frac{\Omega}{2} \{ (\alpha \mp 1) \sin[p(1-\lambda)t] + (\alpha \pm 1) \sin[p(1+\lambda)t] \} \end{aligned} \quad (8-5)$$

The upper sign in the above expression applies for Max I vehicles and the lower

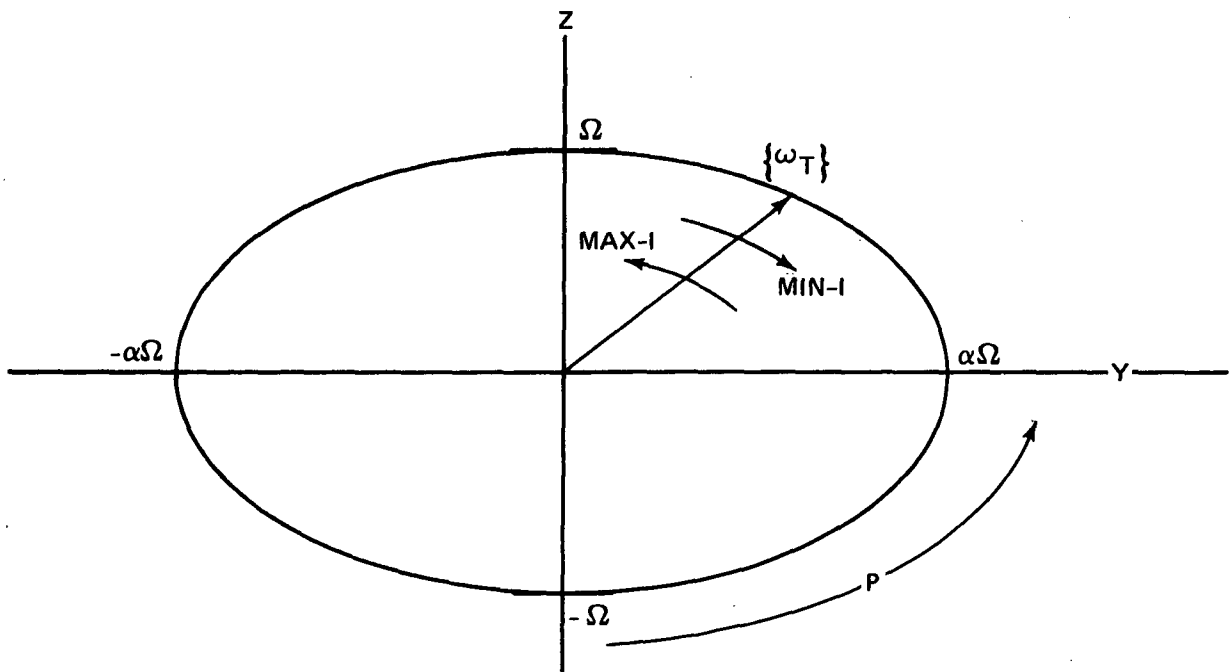


Figure 8-1. Wobble Vector Projected on Rotor Coordinates

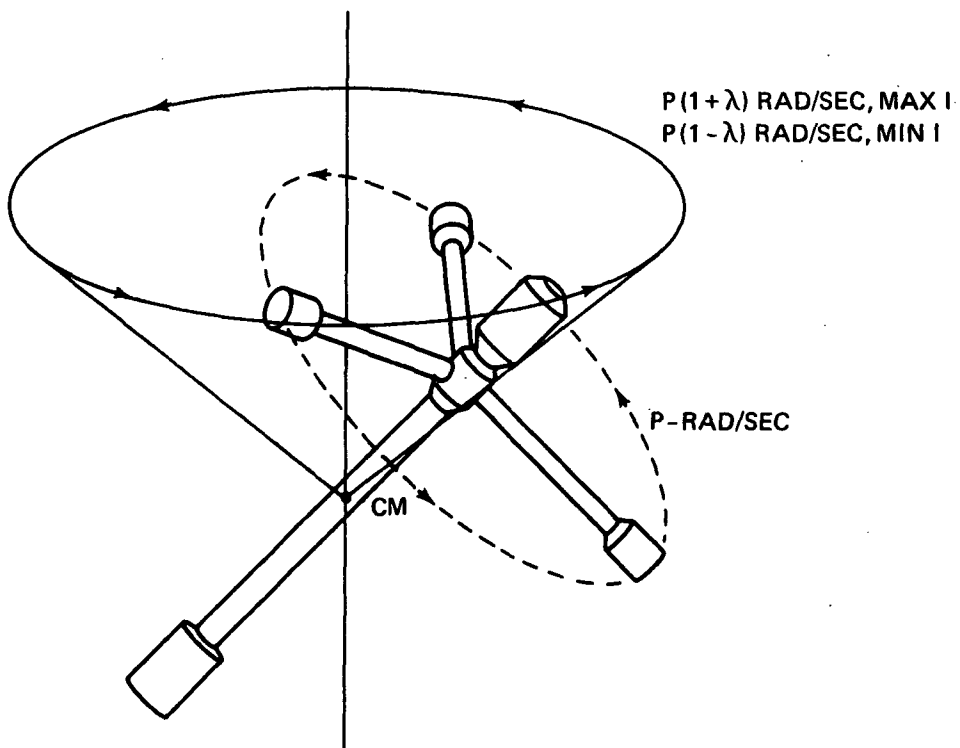


Figure 8-2. Inertial Motion Of A Symmetric Spinning Space Station

sign applies for Min I vehicles. The general expression (8-5) consists of the sum of two sinusoids: one with a frequency equal to the spin plus the wobble frequency, and the other frequency equal to the spin minus the wobble frequency. For most spinning satellites α is close to unity and, in fact, the deviation from unity first becomes appreciable for satellites whose inertia characteristics are close to an "Intermediate Inertia" vehicle. Therefore, the predominate inertial wobble mode for most Max I vehicles is at a frequency equal to spin plus wobble frequency (i.e., $p + \lambda p$). In order to visualize wobble motion let us examine the motion of a symmetric vehicle ($\alpha = 1$) which is characterized by a pure coning motion in inertial space plus the spin motion of the rotor (as shown in Figure 8-2). For an unsymmetric vehicle ($\alpha \neq 1$) small cyclic motions are superimposed on the coning motion shown in Figure 8-2.

When the structure is made flexible and energy dissipation exists then the coning motion, or wobble, will either converge or diverge depending upon the location of the energy absorbing material. The stability of each basic space station was investigated with two sources of structural flexibility:

- Bearing
- Extension Arms (both hub and rotor)

The influence of a flexibly attached shuttle on vehicle stability was also studied.

Section 8.2 presents the results of analyses performed on a vehicle whose only source of flexibility is its bearing. These results allow for the separation of the stability trends due to just a flexible bearing, as well as being representative of total system stability for space stations with extremely stiff extension arms. The results of total system stability investigations is presented in Section 8.4. A description of what are considered the significant mode shapes is included in Section 8.3. This information will allow the reader to visualize the predominate structural motion of the flexible rotor and flexible hub.

8.2 Stability Sensitivity To Bearing Parameters

The bearing assembly, that connects the rotor to the hub, is idealized as in Reference 3, and is described in Section 6.0 of this report. As previously

noted (in Section 6.0), the bearing model permits the inclusion of energy-dissipative material in either the bearing outer race connection to the rotor or the bearing inner race connection to the hub. The nominal values for bearing stiffness and damping have been derived in Section 4.3.2; the effects of variations about these nominal values are investigated.

A characteristic root analysis was performed using the set of unforced Simplified Flexible Body Equations as presented in Reference 3 (Section 4.3.1), and repeated in Figure 8-3 for completeness. The mass properties for each space station, were held constant at their nominal values as presented in Section 4.2. The results of the analysis are as follows:

"Y" Space Station - The "Y" configuration is stable for any stiffness and damping combination which includes damping material located on either the bearing inner race or outer race. The time constant associated with the decay of the wobble mode as a function of bearing stiffness and damping is shown in Figures 8-4 and 8-5 corresponding to outer race damping and inner race damping, respectively. The time constant resulting from using nominal bearing coefficients is 18.6 hr. As seen from Figures 8-4 and 8-5 dissipative material on the inner race has more of an influence on wobble damping than dissipative material on the outer race; that is for the same bearing coefficients the time constant resulting from inner race damping is appreciably less than that of outer race damping.

"T" Space Station - The "T" configuration is unstable for outer race damping for any value of stiffness or damping. The time constant associated with the exponential divergence of the wobble mode is shown in Figure 8-6 as a function of bearing stiffness and damping. The time constant for divergence corresponding to nominal bearing parameters is 5.8 hrs.

The "T" station with inner race damping is either stable or unstable depending upon the amount of damping and corresponding stiffness. A stability boundary is constructed and is shown in Figure 8-7. In general, a low bearing stiffness in combination with a high damping coefficient will improve the possibility of vehicle stability. Figure 8-8 presents the time constant associated with either an exponential decay or divergence as a function of the bearing parameters.

$$I_2 \dot{W}_2 = -(I_1 - I_3) p W_3 - E_B \dot{\Theta}_2 - K_B \Theta_2 + n_H E_B p \Theta_3$$

$$I_3 \dot{W}_3 = (I_1 - I_2) p W_2 - E_B \dot{\Theta}_3 - K_B \Theta_3 - n_H E_B p \Theta_2$$

$$\dot{h}_2 = p h_3 - I_1 p W_3$$

$$\dot{h}_3 = -p h_2 + I_1 p W_2$$

$$Q \dot{\Theta}_2 = -h_2 + (I_2 + Q) W_2 + Q p \Theta_3$$

$$Q \dot{\Theta}_3 = -h_3 + (I_3 + Q) W_3 - Q p \Theta_2$$

where:

$$Q = I_H + M_H a_H^2 + M_R a_R^2$$

I_1, I_2, I_3 = moement of inertia of rotor (spin is 1 axis)

I_4 = transverse moment inertia of hub (euqations assume hub symmetry)

M_R, M_H = mass of rotor, hub

a_R, a_H = Distance for combined system CM to CM of rotor, hub

p = spin rate

$\{h\}$ = angular momentum of entire system

$\{O\}$ = ordered rotations from hub to rotor

ω_2, ω_3 = transverse angular velocity components of rotor

K_B = Bearing Stiffness

E_B = Bearing Damping

n_H = $\begin{cases} 0 & \text{- Damping material on bearing outer race} \\ 1 & \text{- Damping material on bearing inner race} \end{cases}$

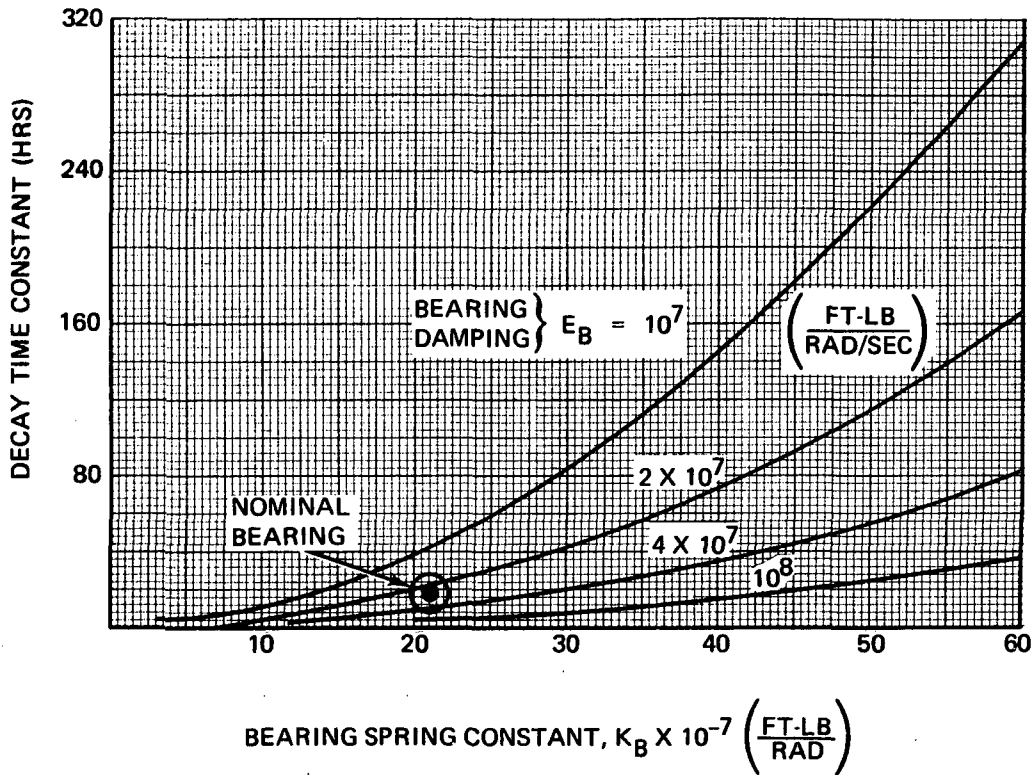


Figure 8-4. Decay Time Constant ^① for "Y" Outer Race Bearing Damping

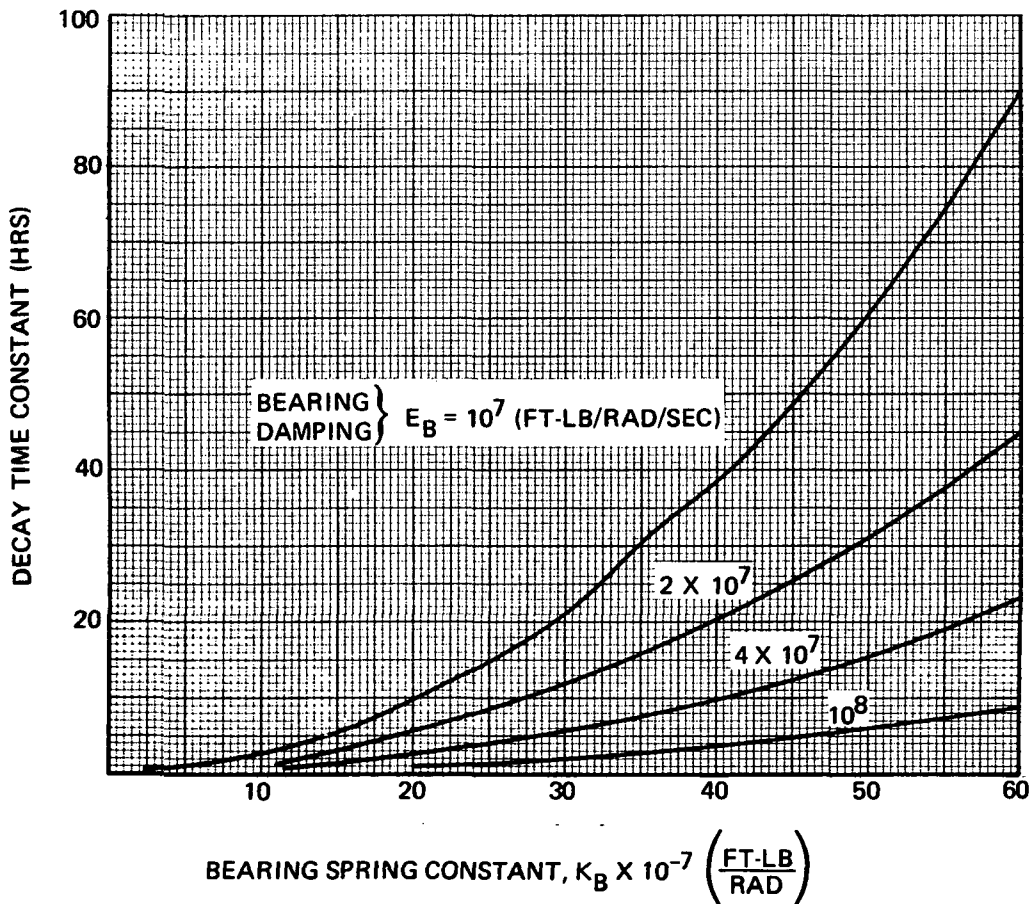


Figure 8-5. Decay Time Constant ^① For "Y" Inner Race Bearing Damping

^① TIME CONSTANT (τ) AS IN, $e^{-t/\tau}$

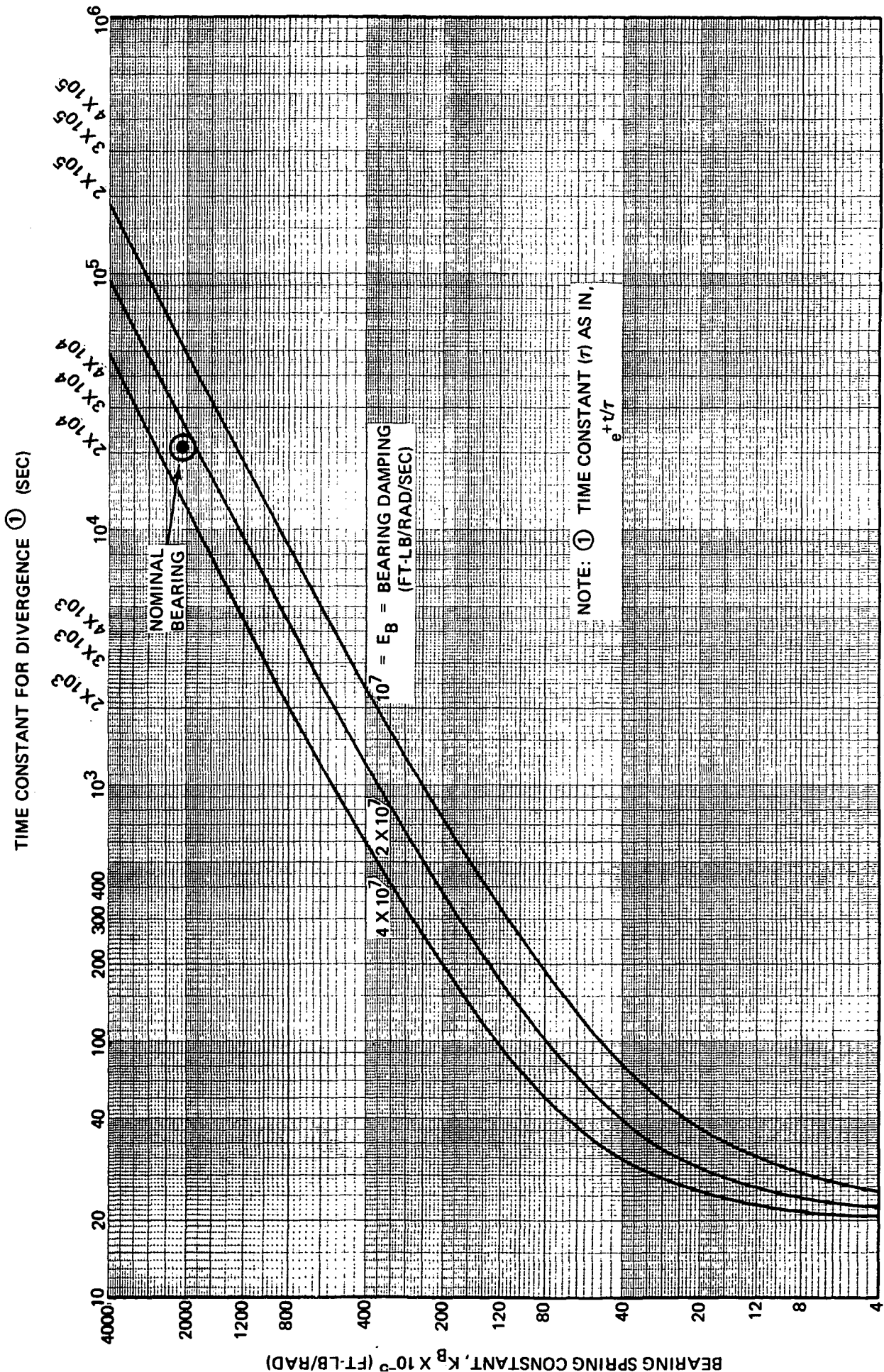


Figure 8-6. Time Constant For Divergence For "T" - Outer Race Bearing Damping

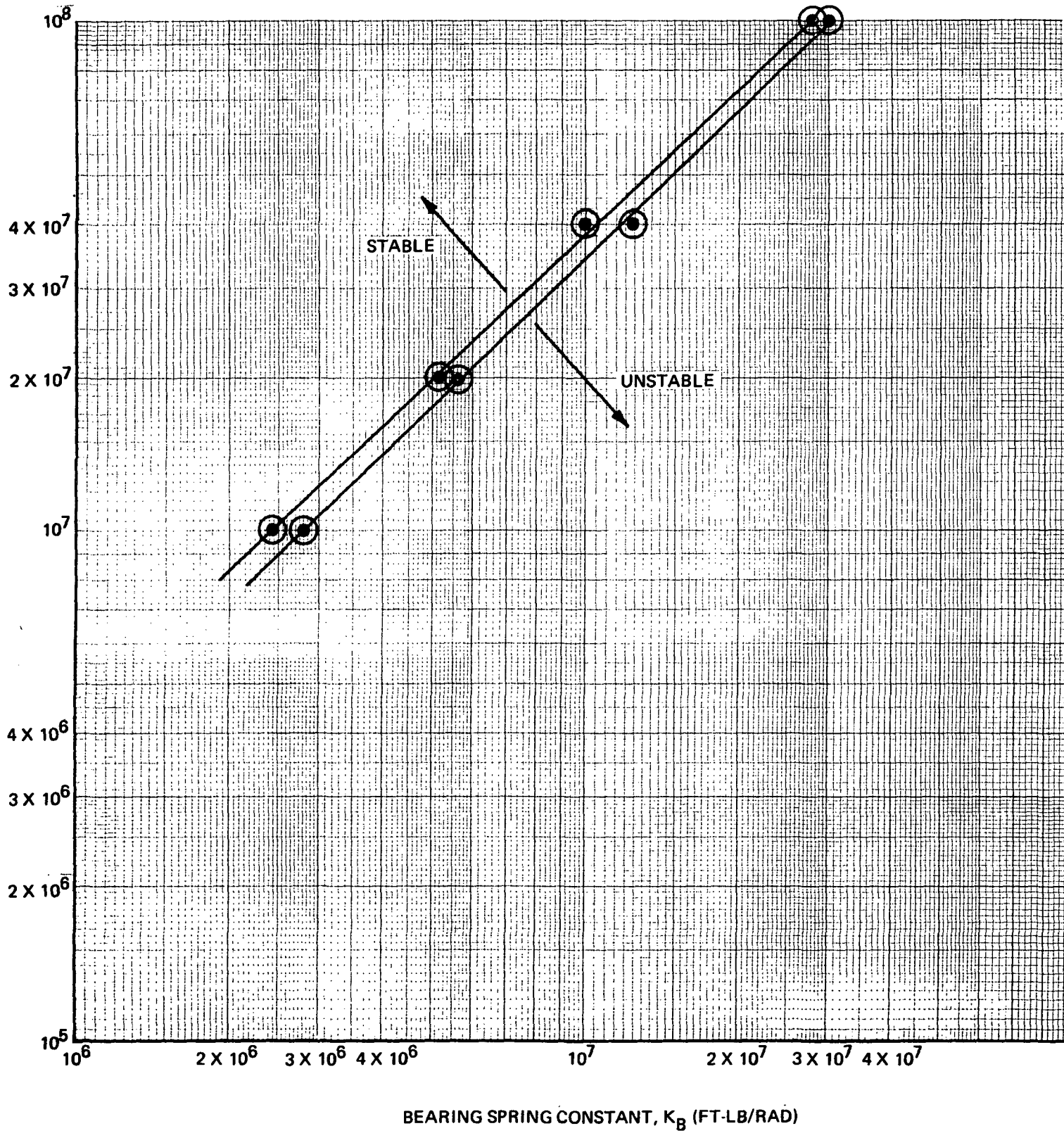


Figure 8-7. Stability Boundary For "T" - Inner Race Bearing Damping

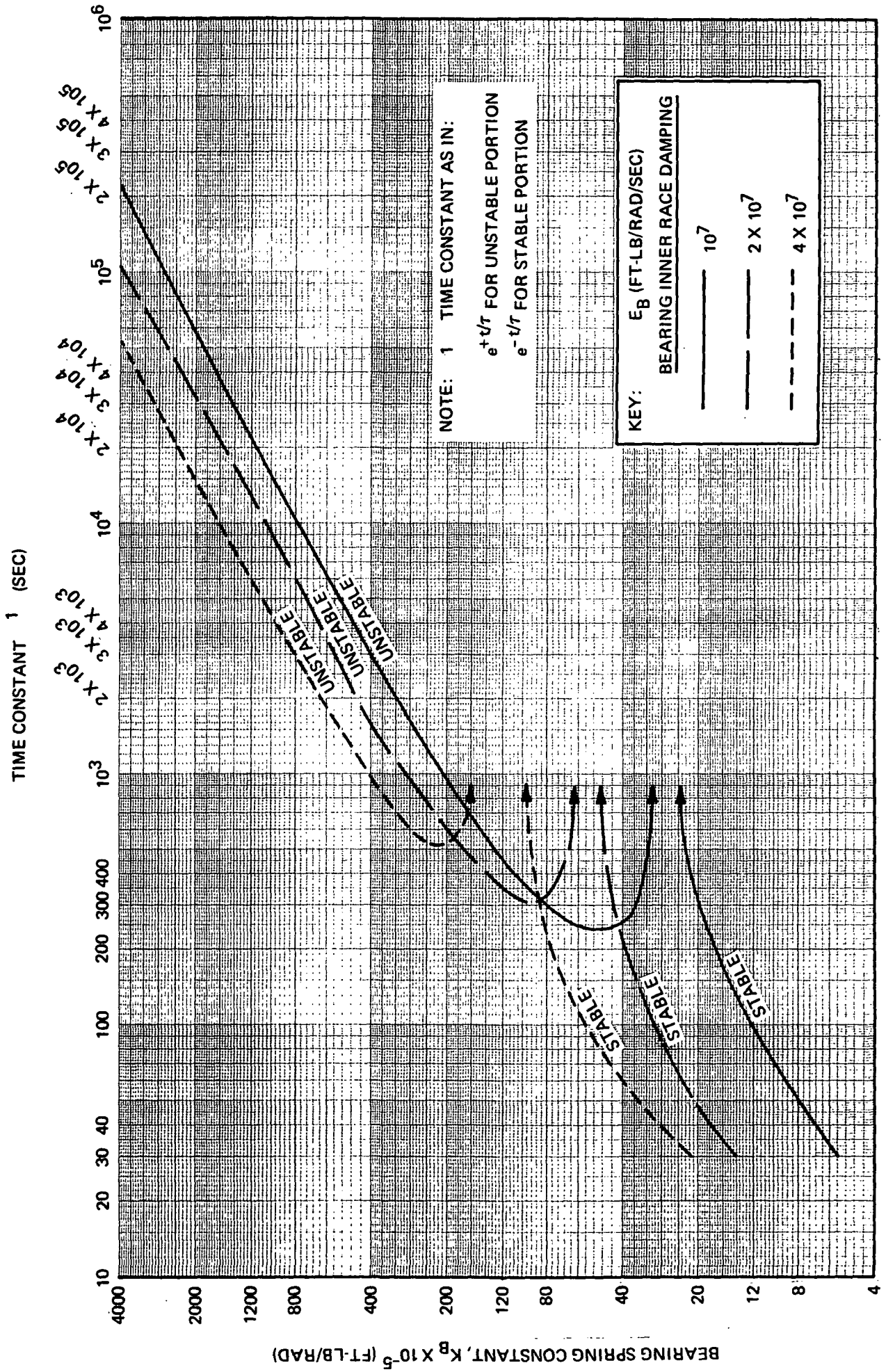


Figure 8-8. Time Constant For "T" - Inner Race Bearing Damping

Until this point in the report the rotational dynamics have been described by words, equations, illustrations and characteristic roots. A useful tool for examining vehicle dynamics is the Time History Simulation and the resulting time history plots. However, before proceeding with analyzing the time history plot resulting from the General Flexible Body simulation (which includes extension arm flexibility), let us first examine the time history plots from a simulation whose only flexible member is the bearing. Figures 8-9 and 8-10 are time histories of the transverse angular rates for the nominal "Y" and "T" configurations, respectively. The "Y" simulation was initialized with a transverse component of angular momentum in the hub while the "T" run was initialized with an angular momentum component in the rotor. The high frequency flexible mode, representing relative motion between the rotor and hub, is seen to damp out completely within the first 20 seconds. The remaining oscillatory motion is wobble. Although unobservable on these plots, the wobble motion of the "Y" station is converging while the wobble of the "T" is diverging. Due to the size of the time constant, convergence or divergence can only be observed by examining the printed data of the wobble magnitude $|\{\omega_T\}|$.

It is interesting to note that since α for "Y" is closer to one, a predominant frequency equal to spin plus wobble frequency ($p + \lambda p$) can be observed when viewing hub transverse rates. In contrast, the "T" station is close to being an "Intermediate Inertia" configuration (i.e., $\alpha = 5.85$) and as a result the hub rates exhibit two frequency components ($p \pm \lambda p$) of relatively equal weights. These time history plots will be used as a basis of comparison when examining the time histories of vehicles with additional flexible members (refer to Section 8.4).

It should be noted that when a shuttle is rigidly attached to the station, the "Y" configuration as well as "T" are "Min I". As a result, both stations when combined with the shuttle are unstable for the condition of nominal bearing flexibility provided there is no other source of energy dissipation.

FIGURE 8.9 "Y" WITH BEARING FLEXIBILITY

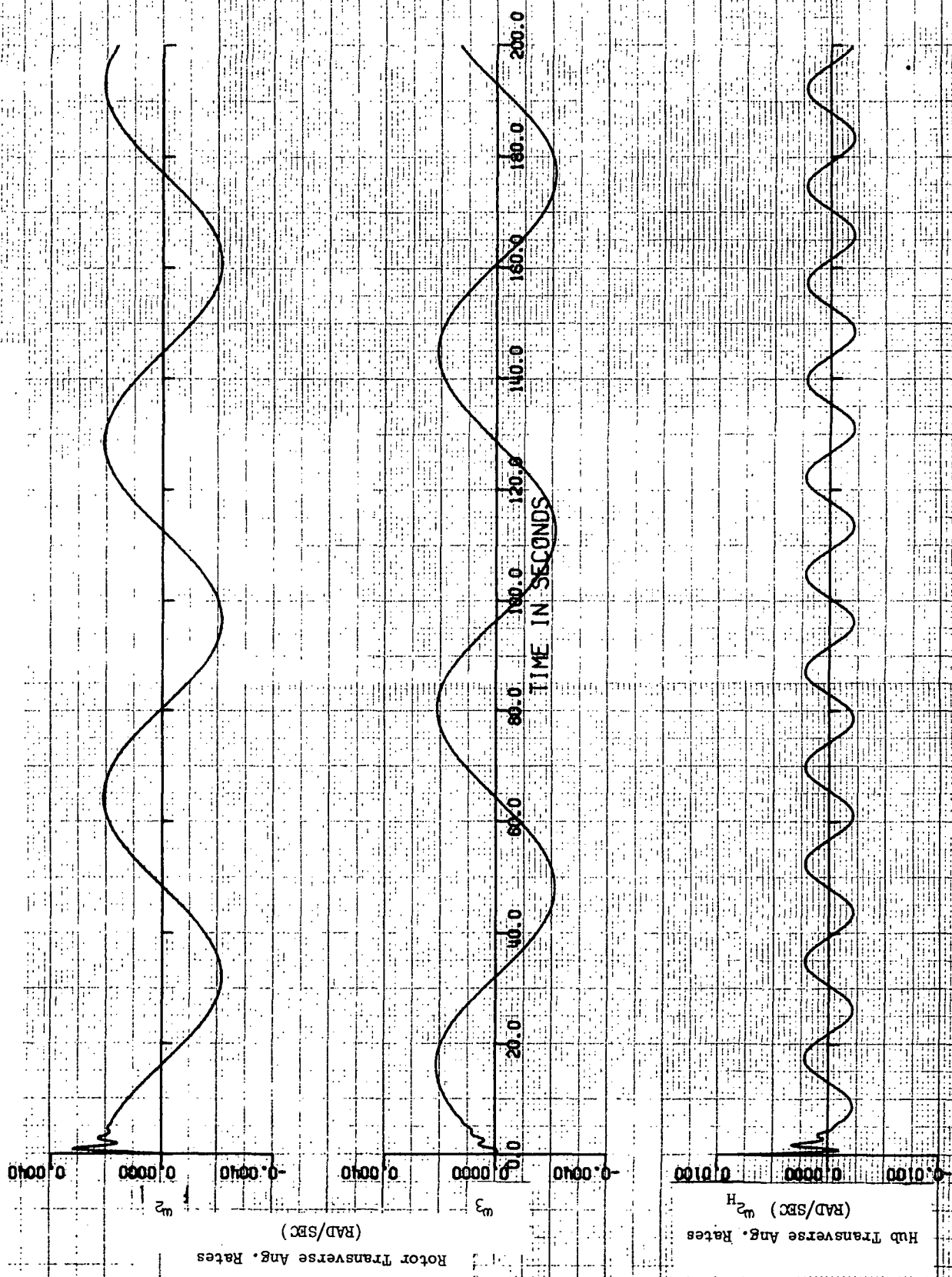


FIGURE 8.9 (Con't)

Hub Transverse Ang. Rates
 ω_{CH} (RAD/SEC)

0.0040 0.0000 -0.0040

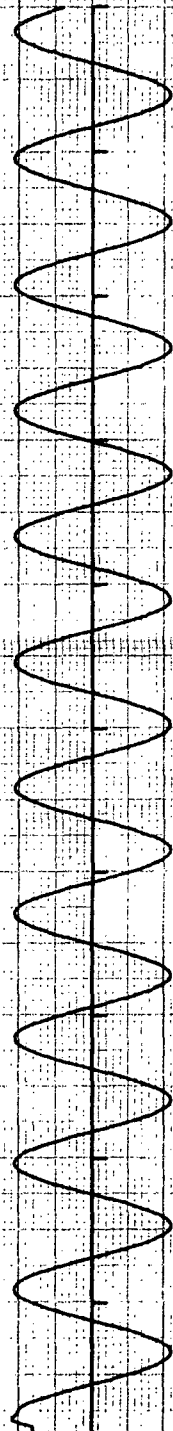
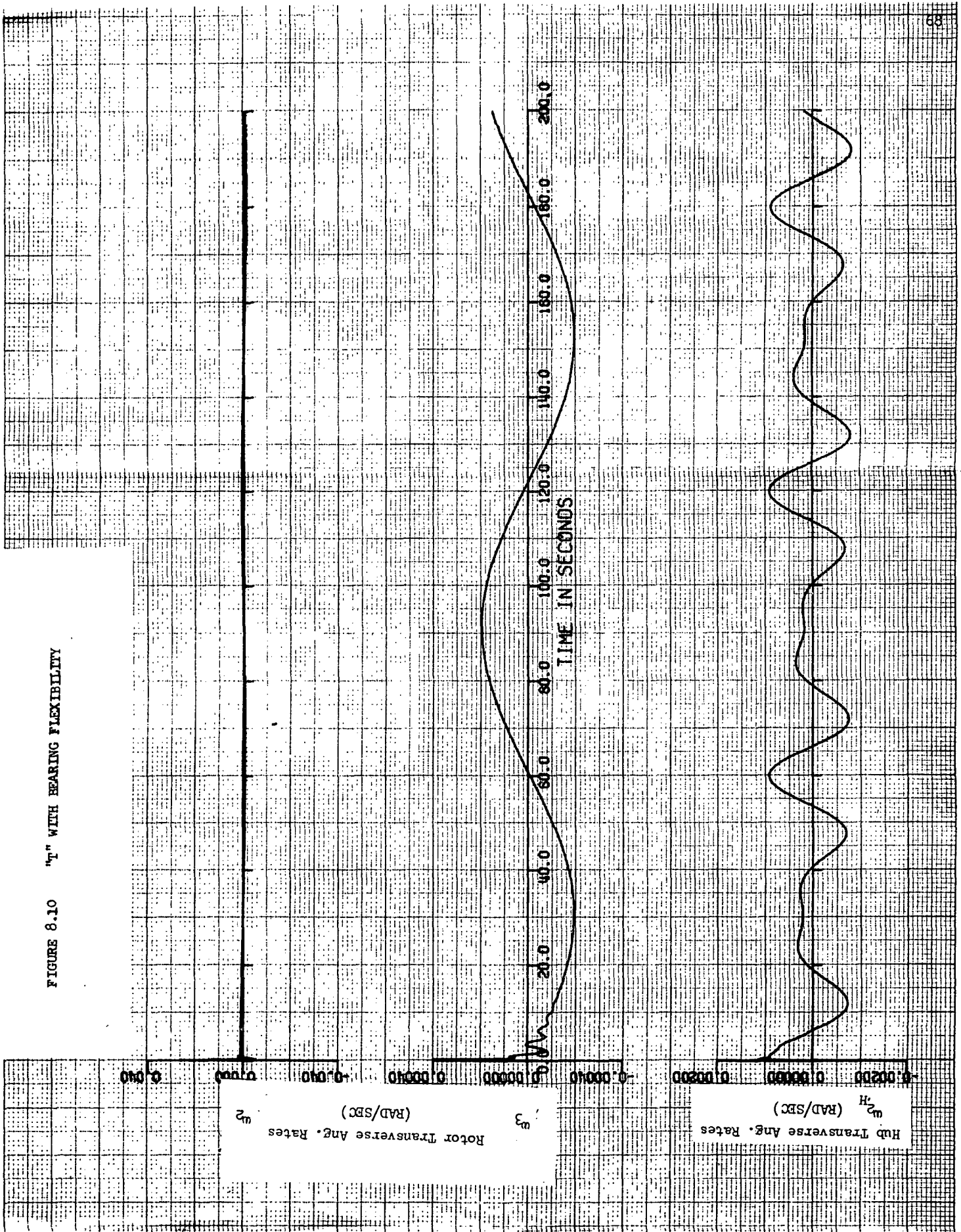


FIGURE 8.10 "T" WITH BEARING FLEXIBILITY



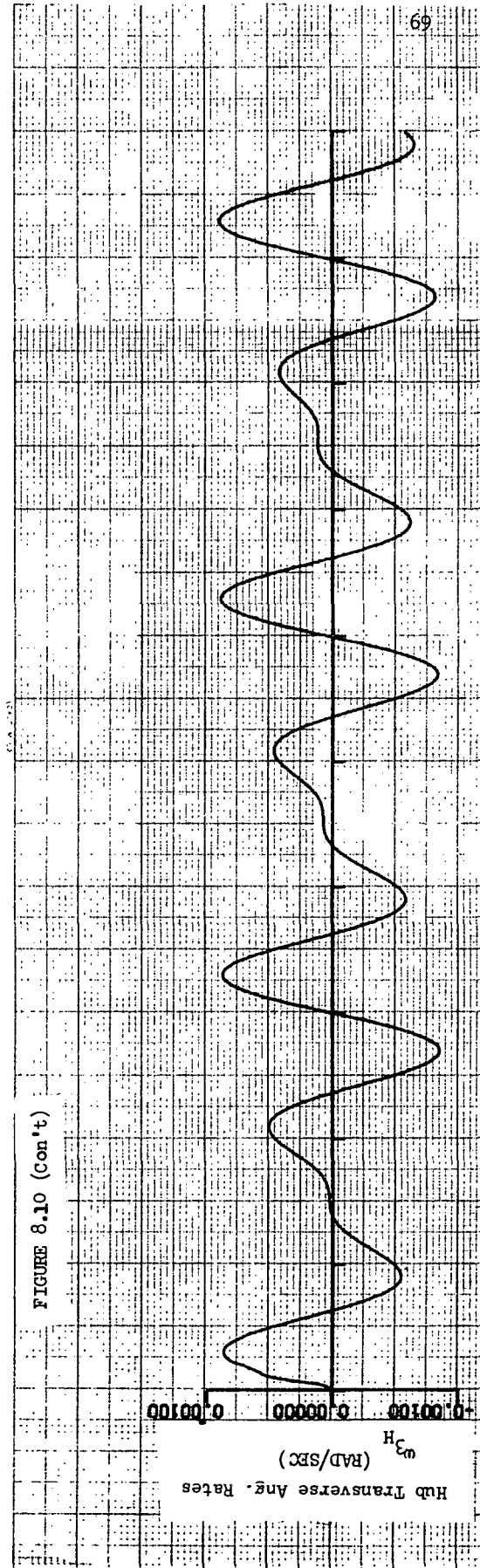
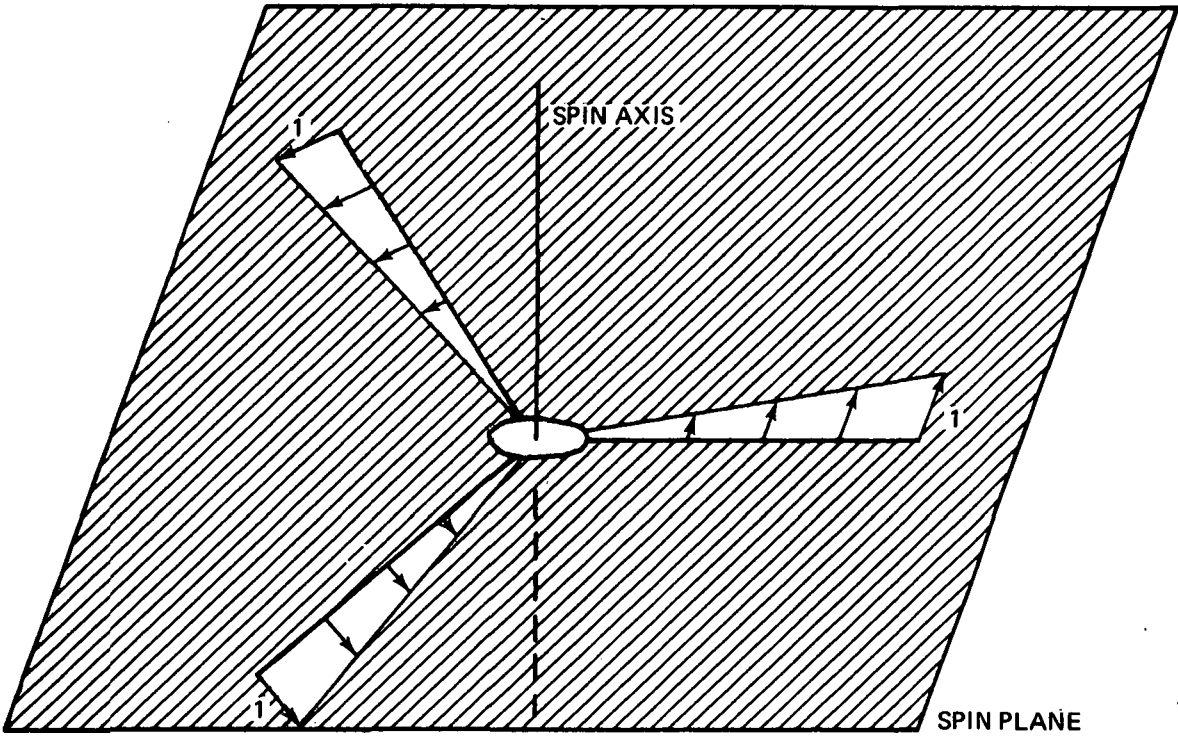


FIGURE 8.10 (Con't)

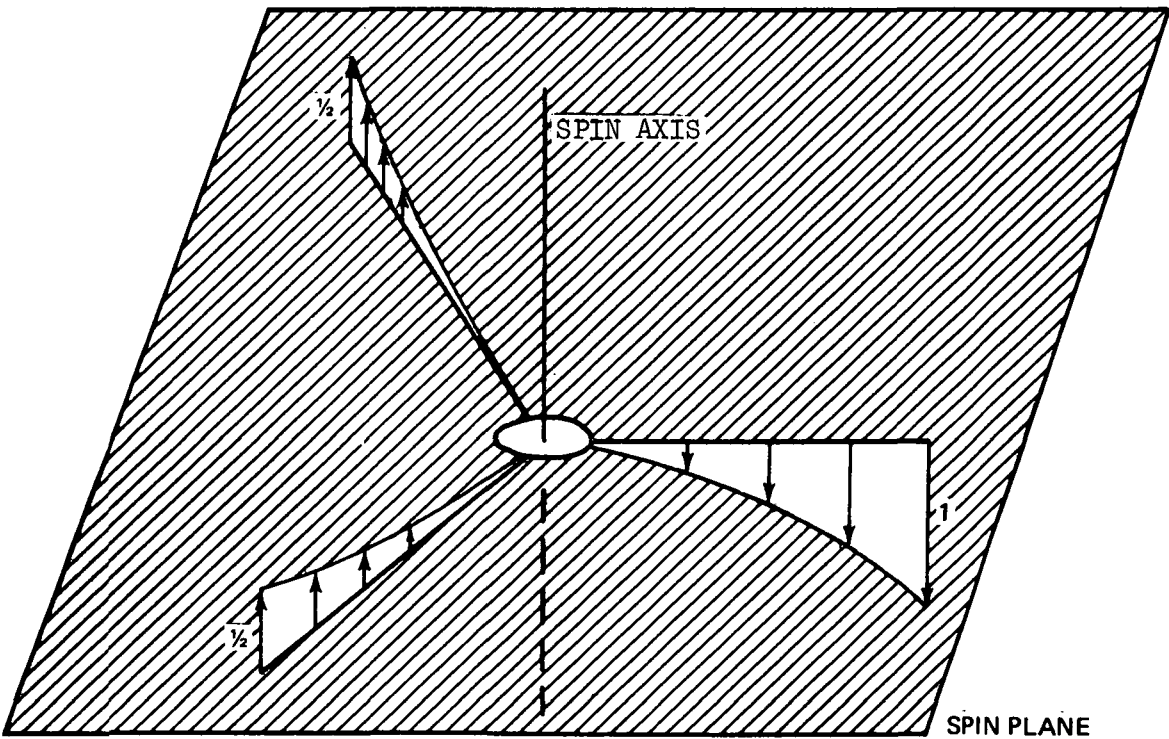
Hub Transverse Ang. Rates
 ω_H (RAD/SEC)

8.3 Significant Mode Shapes

Figures 8-9 through 8-15 show the rotor and hub modes used for the majority of simulation studies. The listed modal frequencies correspond to the "Y" space station with baseline properties. Six rotor modes and two hub modes were used for each vehicle. For the rotor, the predominant motions are i) in plane bending, ii) out-of-plane bending, and iii) torsion. Torsion in the T-rotor occurs independently of bending and is easy to visualize (Figures 8-14a and b); for the Y-rotor, however, some torsion accompanies out-of-plane bending and is difficult to show; e.g. there is some torsion in arms two and three in Figure 8-9(b). Predominant hub motions are bending in each of the two transverse planes. As was expected, axial extension of the beams occurs at very high frequency and is not present in any of the modes shown.

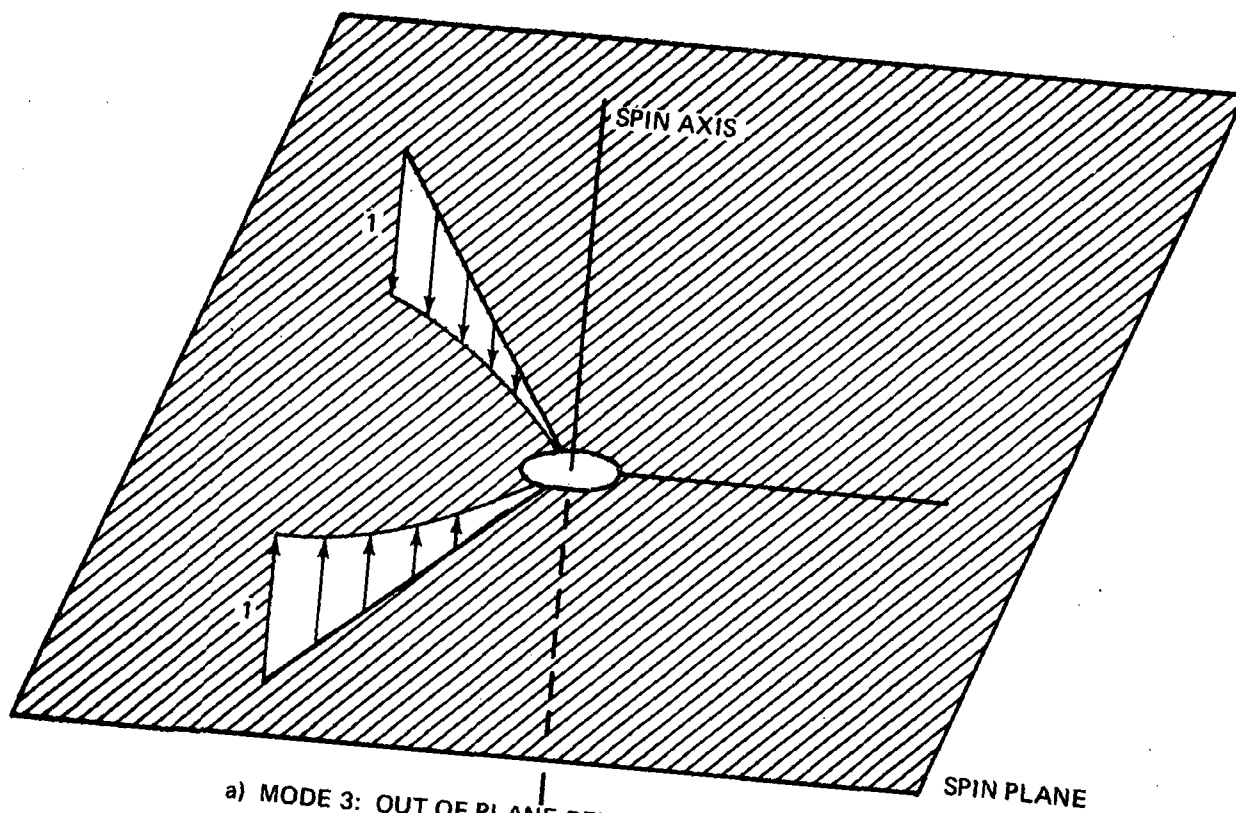


a) MODE 1: RIGID BODY MODE, IN PLANE ($f = 0$)

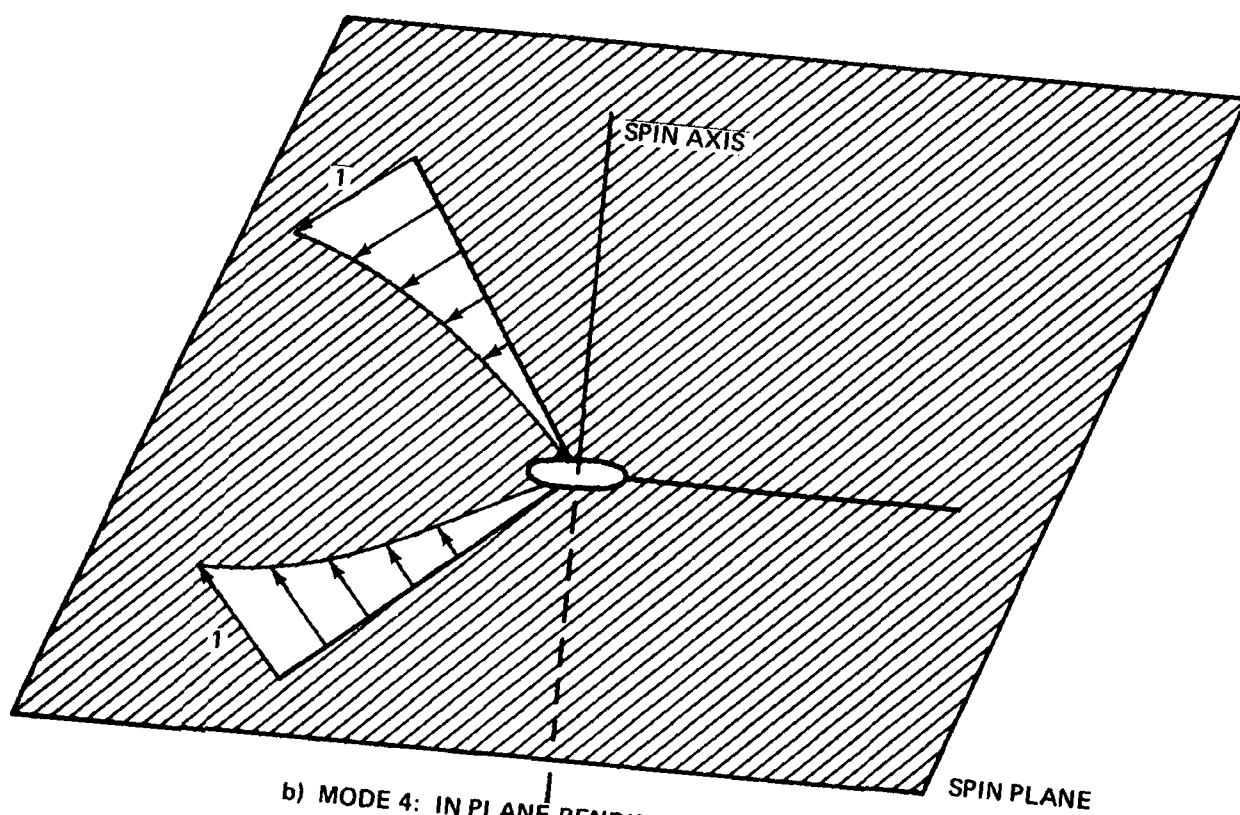


b) MODE 2: OUT OF PLANE BENDING ($f = 0.12881$ CPS)

Figure 8-9. Y Rotor Modes

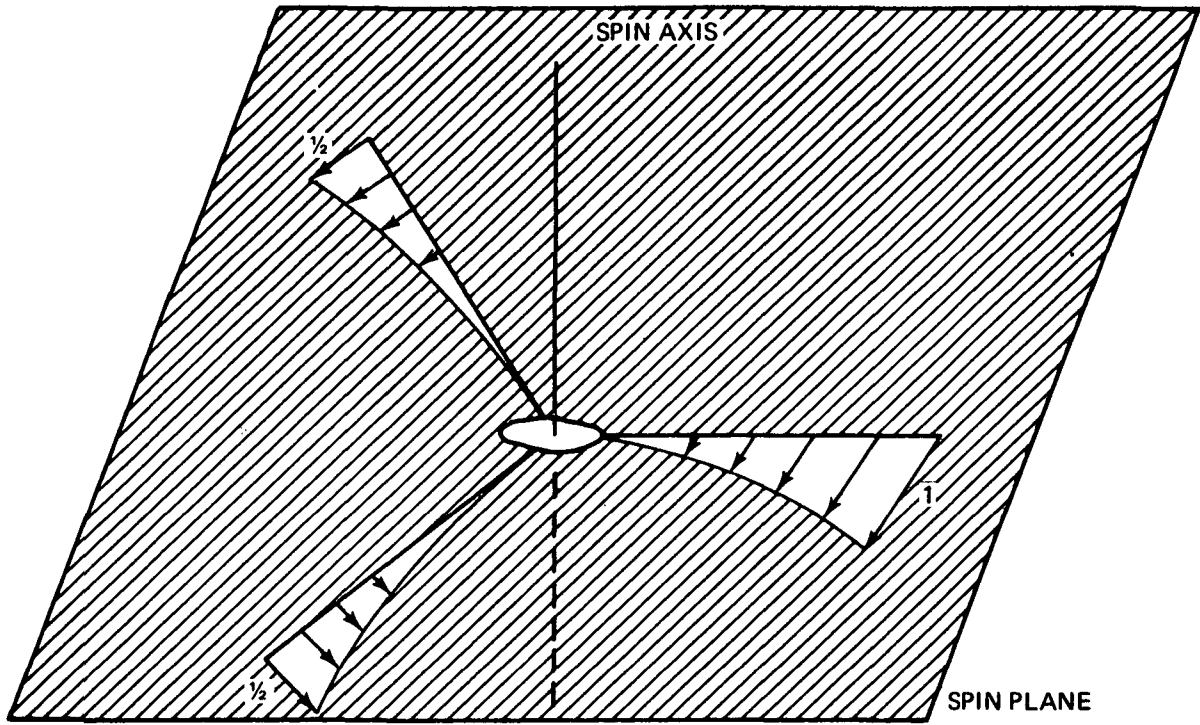


a) MODE 3: OUT OF PLANE BENDING ($f = 0.12885$ CPS)

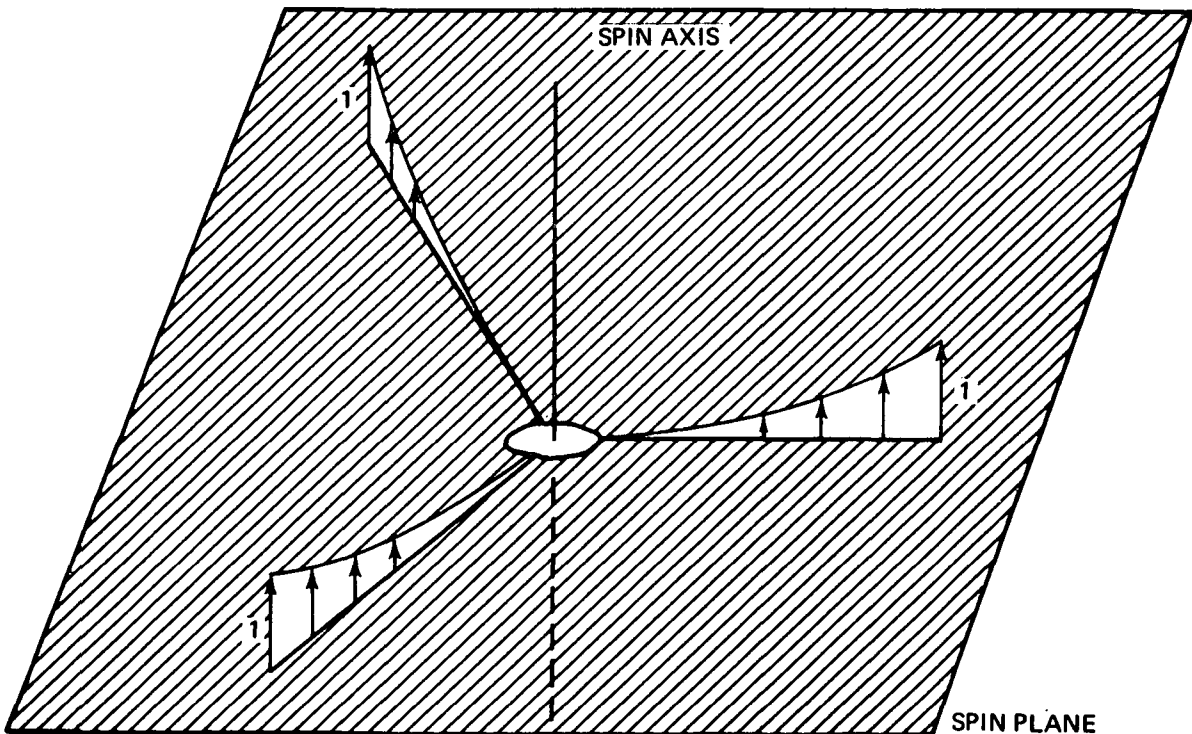


b) MODE 4: IN PLANE BENDING ($f = 0.14138$ CPS)

Figure 8-10. Y Rotor Modes (Cont.)

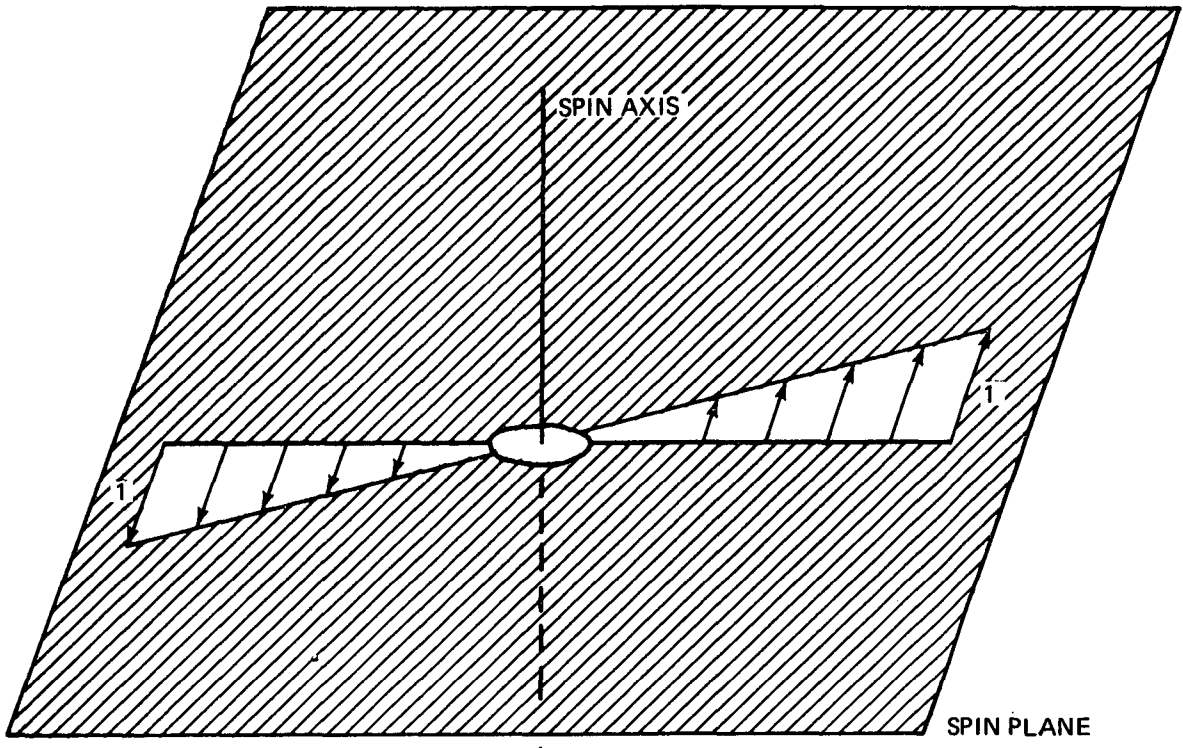


a) MODE 5: IN PLANE BENDING ($f = 0.14155$ CPS)

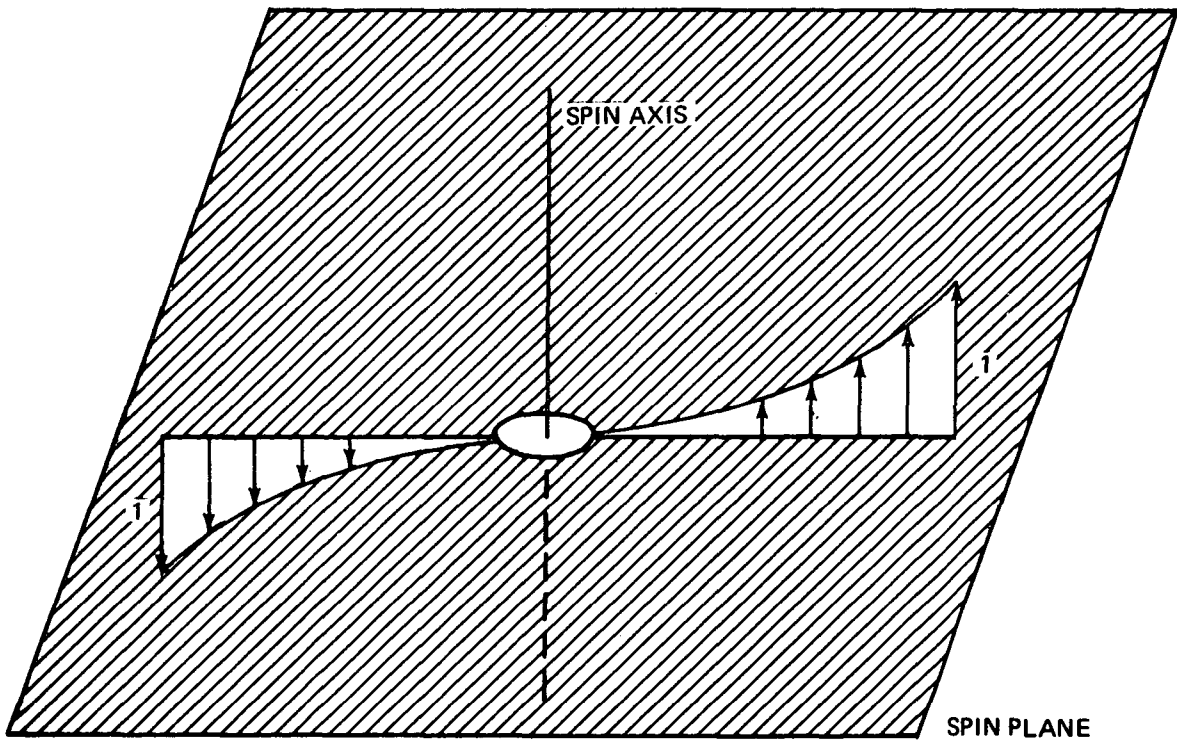


b) MODE 6: OUT OF PLANE BENDING ($f = 0.14196$ CPS)

Figure 8-11. Y Rotor Modes (Cont.)

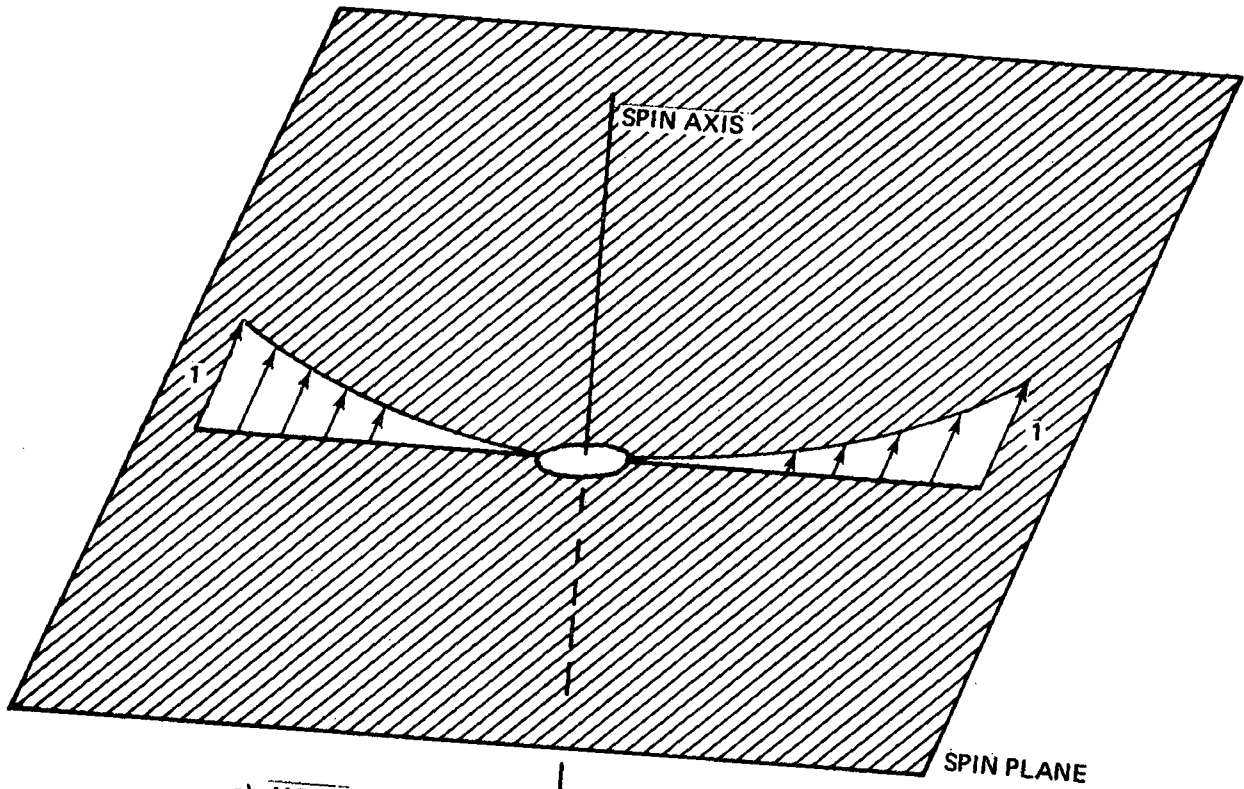


a) MODE 1: RIGID BODY MODE, IN PLANE ($f = 0$)

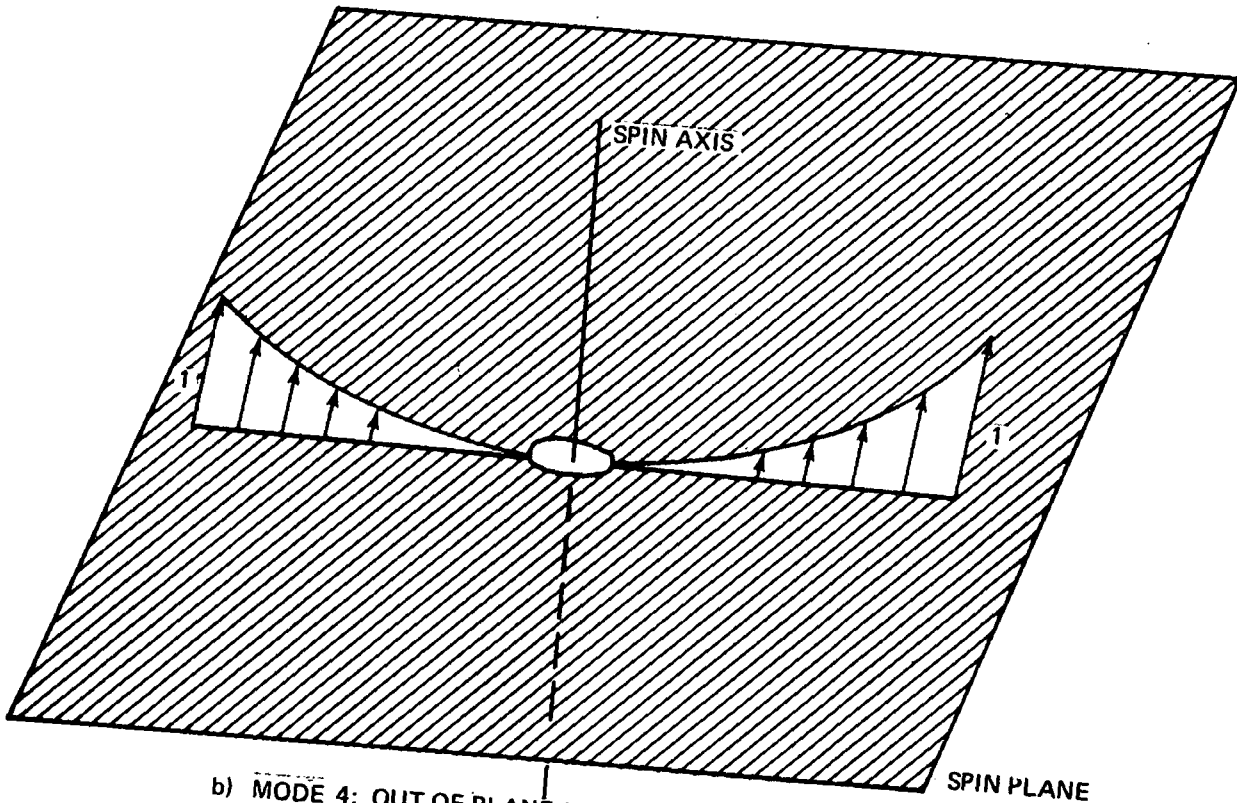


b) MODE 2: OUT OF PLANE BENDING ($f = 0.13007$ CPS)

Figure 8-12. T Rotor Modes

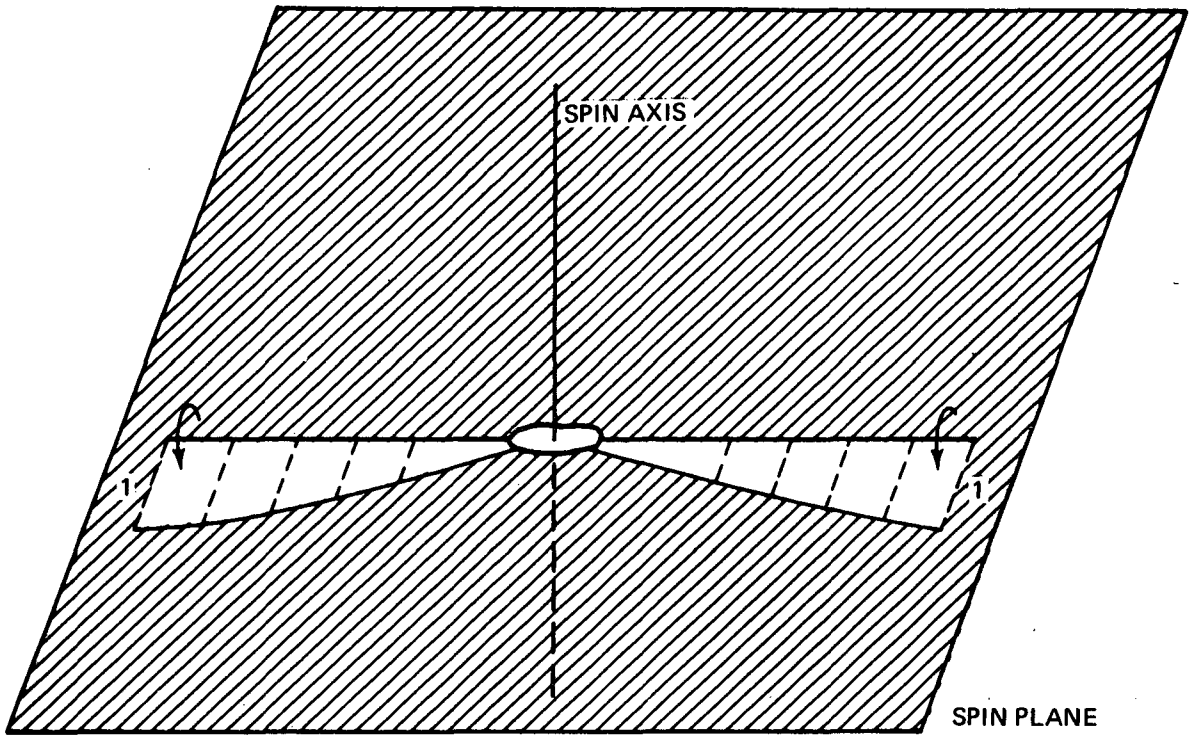


a) MODE 3: IN PLANE BENDING ($f = 0.13719$ CPS)

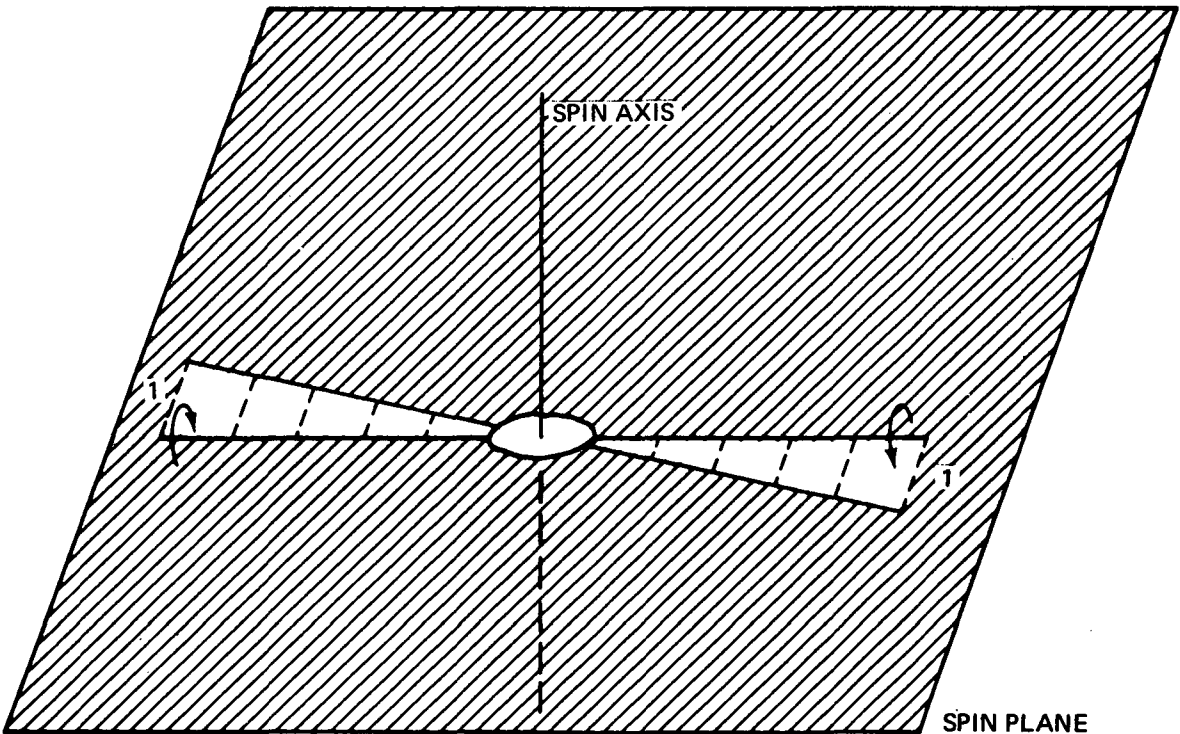


b) MODE 4: OUT OF PLANE BENDING ($f = 0.14816$)

Figure 8-13. T Rotor Modes (Cont.)



MODE 5: TORSION ($f = 0.39320$ CPS)



MODE 6: TORSION ($f = 0.41336$ CPS)

Figure 8-14. T Rotor Modes (Cont.)

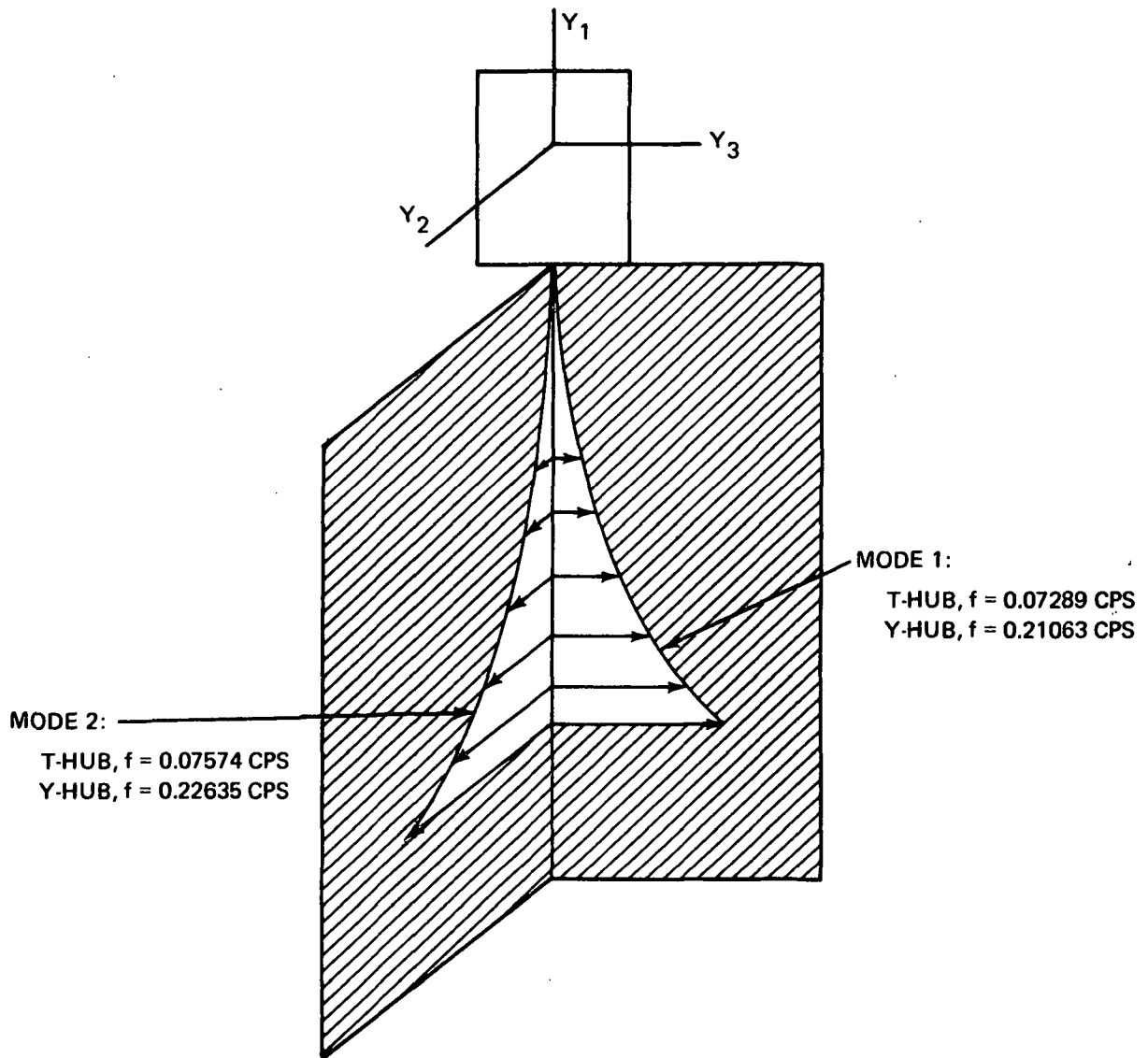


Figure 8-15. Hub Modes (T And Y)

8.4 General Flexible Body Dynamics

The flexible body dynamics of both the "T" and "Y" configurations were investigated using the General Flexible Body Simulation - Stage II. This simulation, specifically developed for this study program, is described briefly in Section 7.0, and is presented in more detail in Part III of this report in the form of a User's Guide. The simulated equations of motion are presented in Section 6.0. These equations have been formulated to include the effects of the following flexible structural members: rotor arms, hub, bearing, and shuttle docking mechanism.

An exact solution to the equations of motions is obtained when all the flexible modes are numerically integrated. Unfortunately, such a task would require an unacceptable amount of digital computer time. A good compromise is to find the minimum number of low frequency flexible modes that can represent the vehicle motion with a negligible error. It should be noted that the low frequency modes are considered most important since they generally exhibit larger flexible motions, they don't damp as fast, and they are most troublesome from a control point of view. The approach used to determine the minimum number of significant modes was to systematically perform simulation runs with fewer and fewer modes until significant changes in the dynamics were observed. It was determined that at least 6 rotor modes and 2 hub modes are required for an accurate representation of both "T" and "Y" dynamics. Section 8.3, contains a detailed description of the significant mode shapes.

Stability is determined by observing either the convergence or divergence of the magnitude of the transverse angular rate as viewed in the rotor ring $|\omega_T|$ (i.e., simulation variable WTMM). Prior to examining $|\omega_T|$ some time should be allowed to elapse until the significant flexible motion has damped.

An appreciation of flexible body dynamics is obtained by observing time history traces of several variables that represent the vehicle motion at different points located throughout the space station. Specifically, the time history plots that are included herein are of the following variables:

- ω^X - angular rate of rotor ring
- ω^Y - angular rate of rigid section of hub
- ω^{V3} - angular rate of the hub end mass
- $\omega^{z^{i3}}$ - angular rates of the end mass of the ith rotor arm
- $\dot{\mu}^{c3}$ - linear velocity of the hub end mass relative to the rigid hub section
- $\dot{\mu}^{D^{i3}}$ - linear velocity of the end mass of the ith rotor arm relative to the rotor ring

The stability of both the "T" and "Y" configurations were investigated using stiffness values for the extension arms that ranged from the baseline numbers as presented in Section 4.3.1 to twice the baseline stiffness which will be referred to as nominal, specifically,

RANGE OF EXTENSION ARM STIFFNESS		
	Baseline, Lower	Nominal, Upper
• Bending EI	$9.34 \times 10^8 \text{ lb-ft}^2$	$18.68 \times 10^8 \text{ lb-ft}^2$
• Torsion GJ	$7.47 \times 10^8 \text{ lb-ft}^2$	$14.94 \times 10^8 \text{ lb-ft}^2$

During the stability analysis all other physical vehicle properties were maintained at their nominal values, as presented in Section 4.0. The results of the uncontrolled stability analysis using the General Flexible Body simulation are as follows:

"Y" - Space Station - The initial conditions for all investigations are equivalent to that experienced as a result of internal mass motion (refer to Section 5.2). The resulting time histories shown in Figures 8-16 and 8-17 correspond to the baseline and nominal stiffness conditions, respectively. Since the resolution of the time history plots is not sufficient, the stability was determined by viewing the printed output data. All stiffness conditions investigated resulted in convergence of the "Y" station wobble, or absolute stability. In fact, the addition of dissipative material to the

FIGURE 8.16 "Y" - BASELINE STRUCTURE, UNCONTROLLED

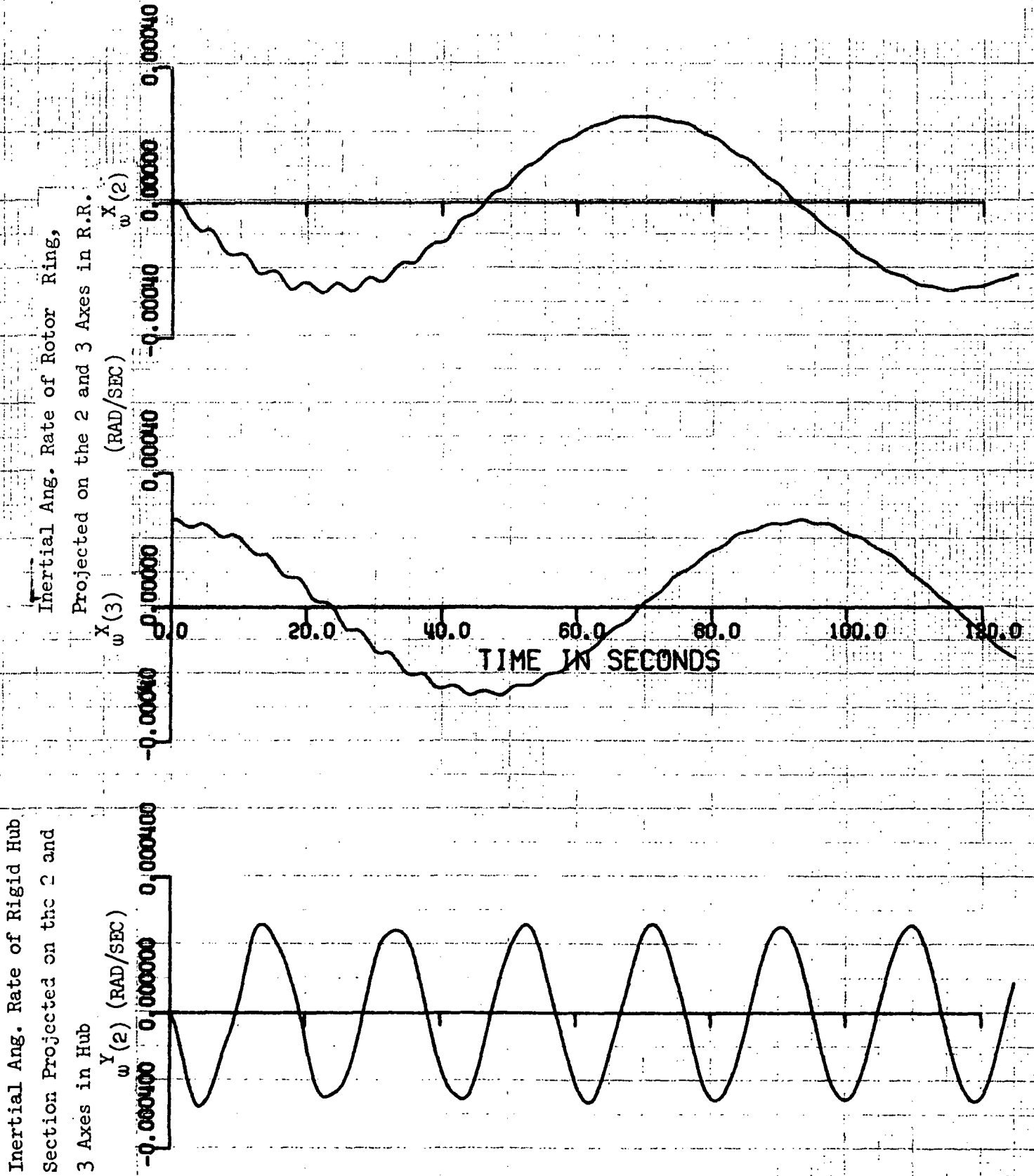
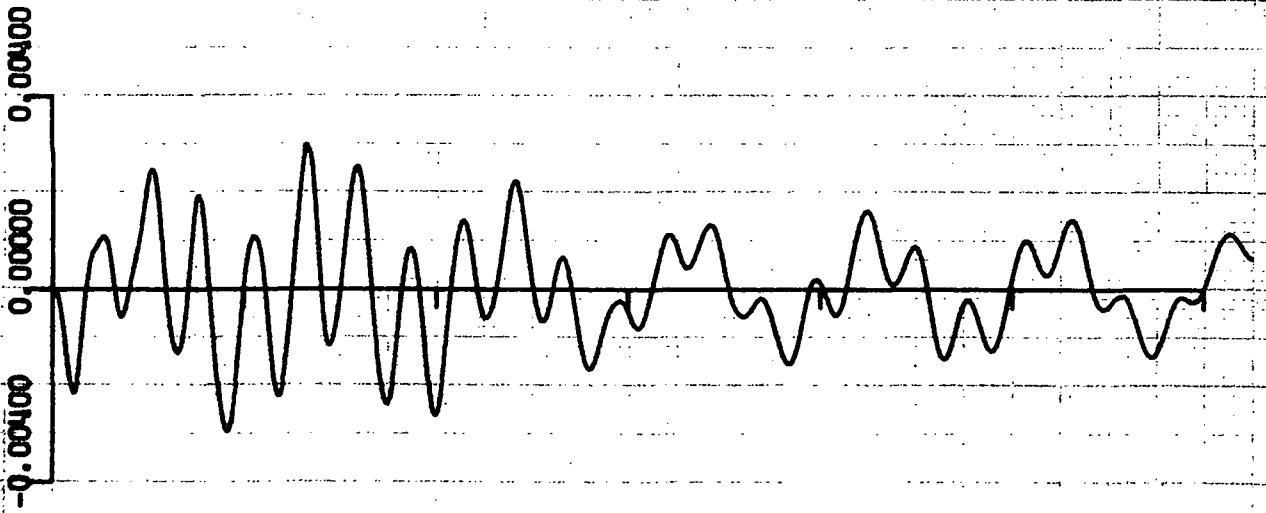


FIGURE 8.16 (Con't)

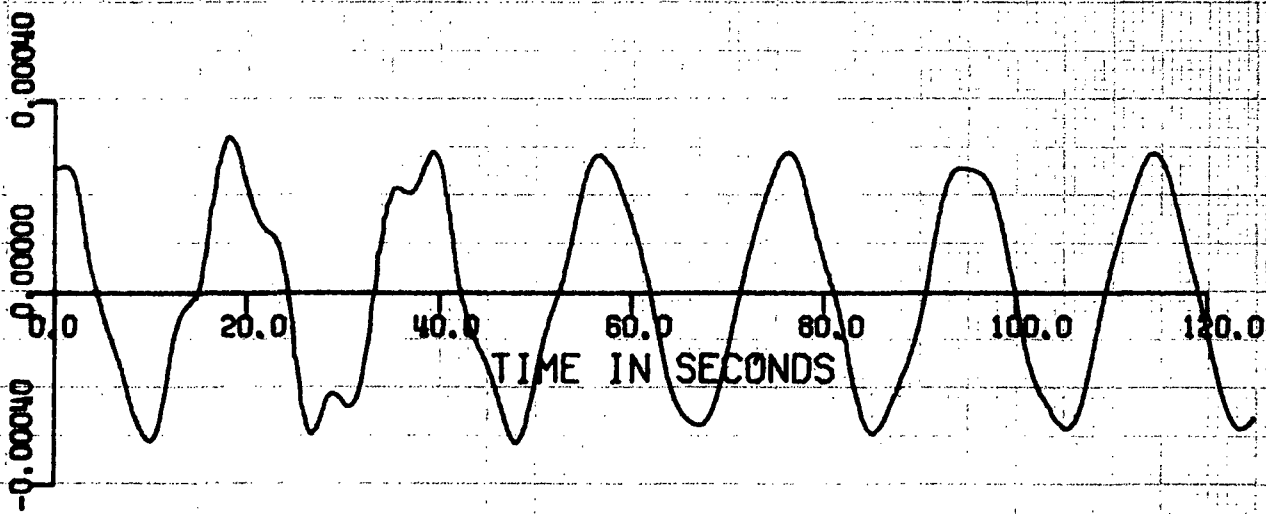
Linear Velocity of the Third Hub
Mass Relative to the Rigid Hub
Section, Projected on the Mass'

2 Axis. $\dot{u}^c(2)$ (FT/SEC)



Inertial Ang. Rate of Third Hub Mass
Projected on the Mass' 3 Axis. $\dot{\omega}^c(3)$

(RAD/SEC)



$\dot{y}^c(3)$

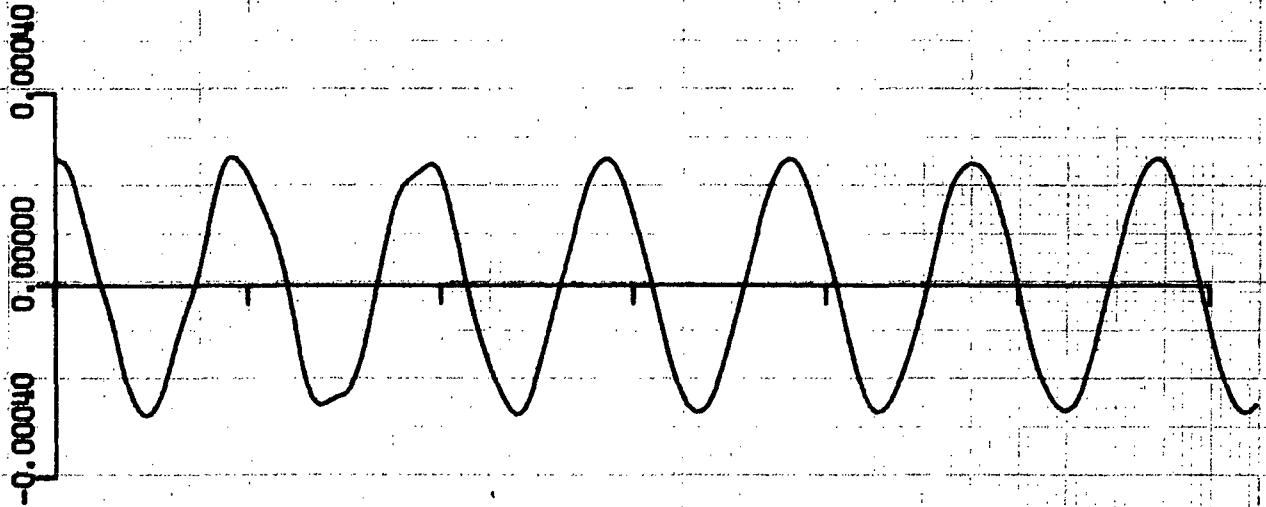
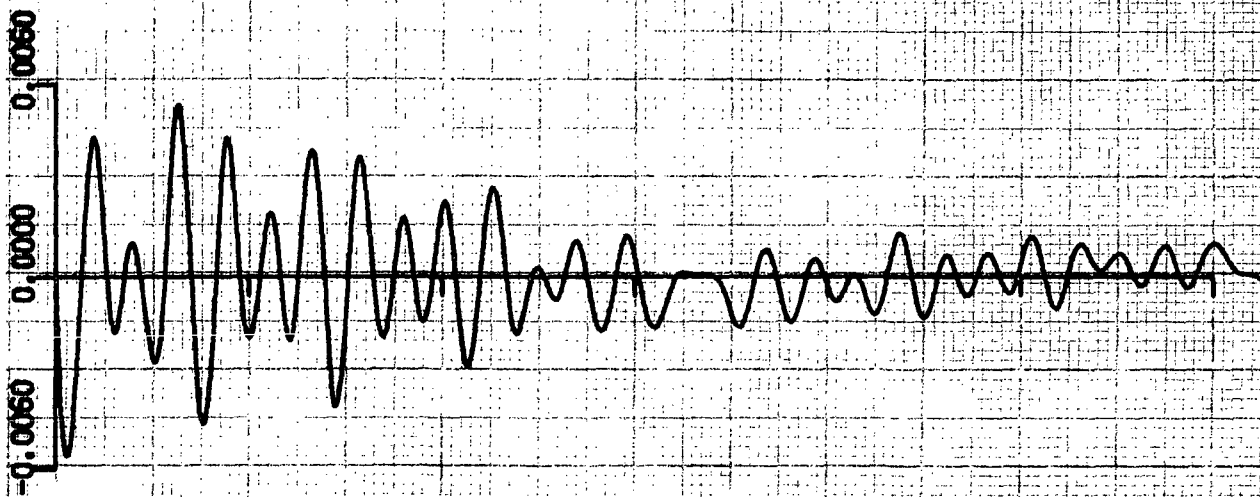


FIGURE 8.16 (Con't)

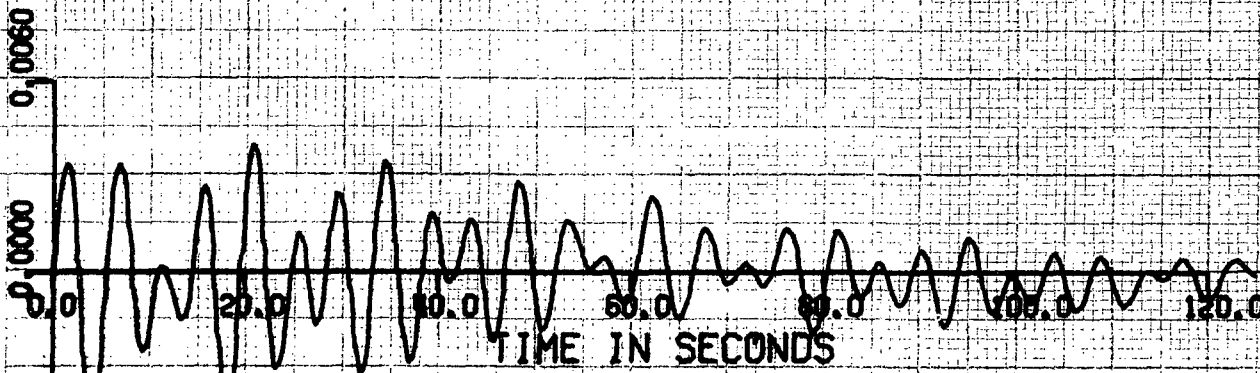
Linear Velocity of Third Rotor
Mass in the First Arm, Relative to
the Rotor Ring, Projected on the
Mass' 1 Axis.

$$\dot{u}^{D13} \text{ (FT/SEC)}$$



Linear Velocity of the Third Rotor
Mass in the Second Arm, Relative to the
Rotor Ring, Projected on the Mass' 1
and 2 Axes.

$$\dot{u}^{D23} \text{ (1)}$$



$$\dot{u}^{D23} \text{ (2)}$$

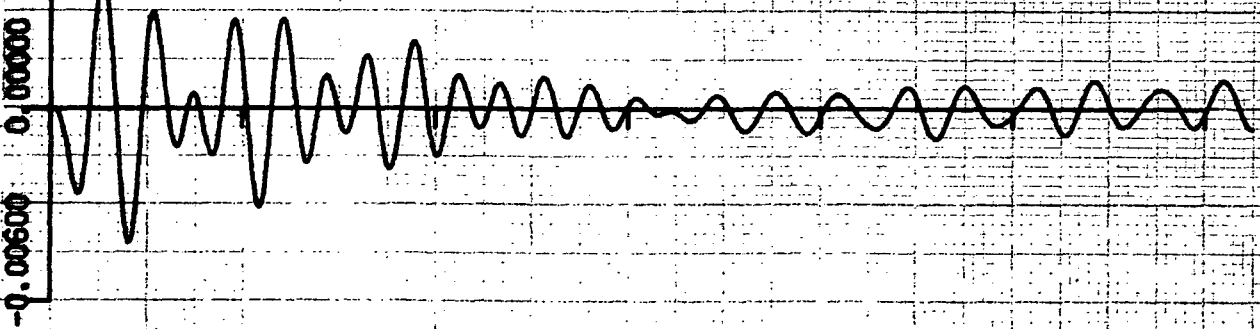


FIGURE 8.16 (Con't)

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis.

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 3 Axis.

Inertial Angular Rate of the Third Rotor Mass in the Second Arm, Projected on the Mass' 2 Axis.

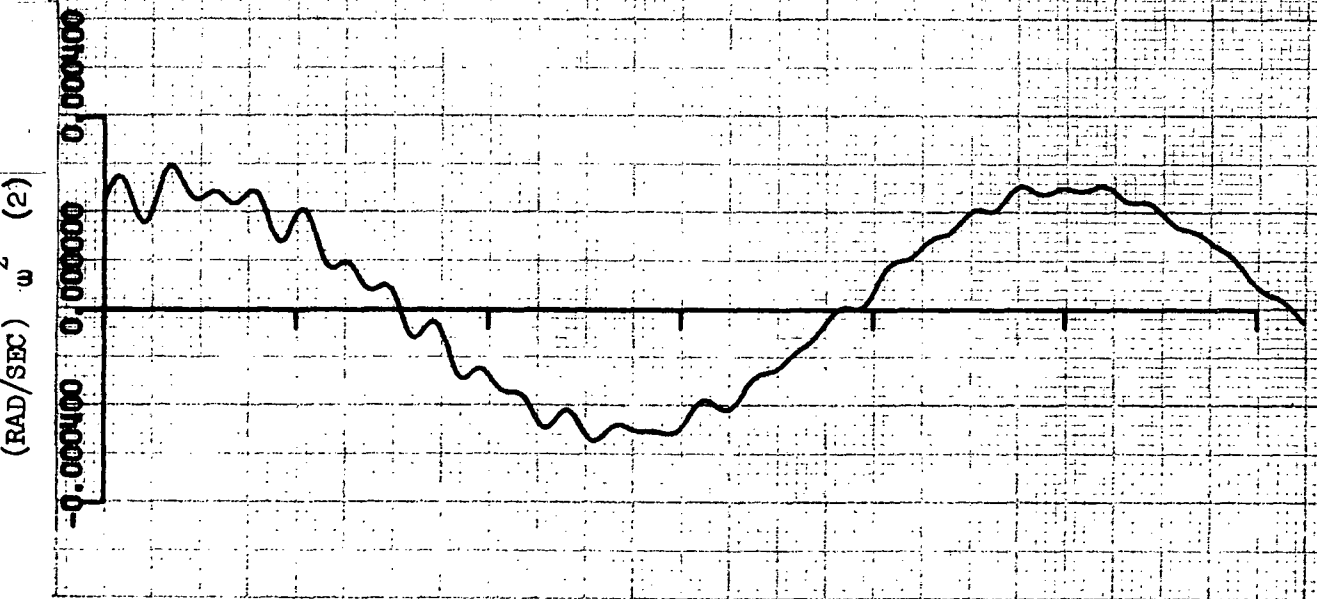
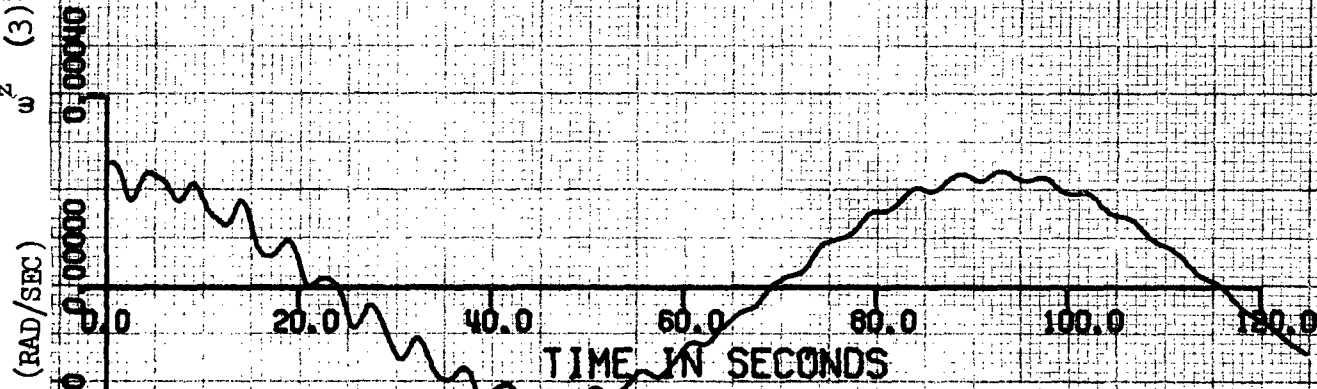
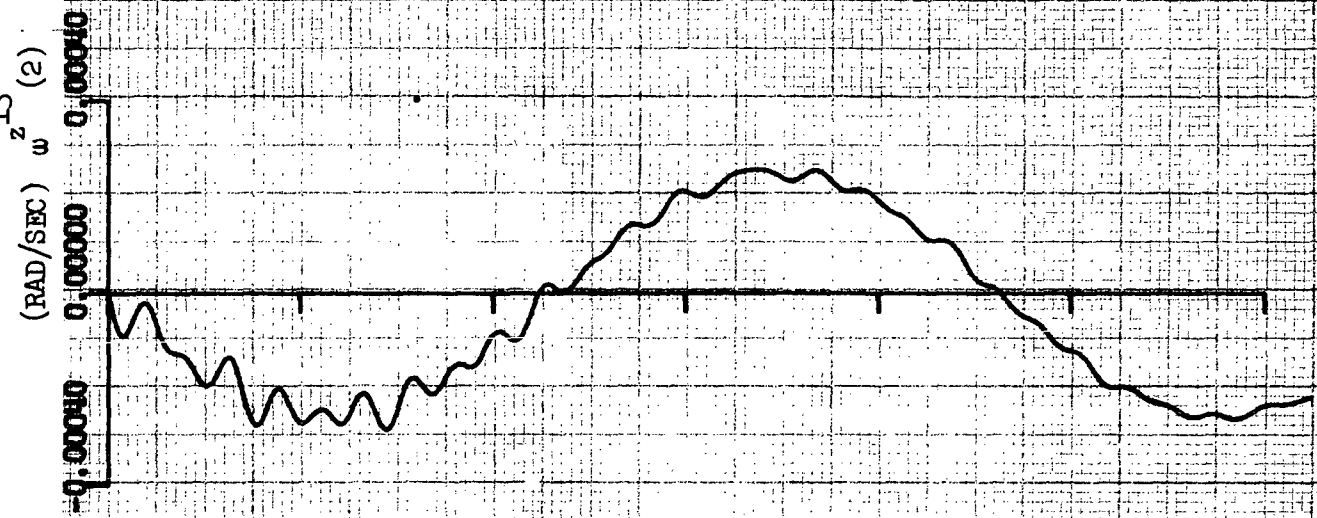


FIGURE 8.17 "Y" - NOMINAL STRUCTURE, UNCONTROLLED

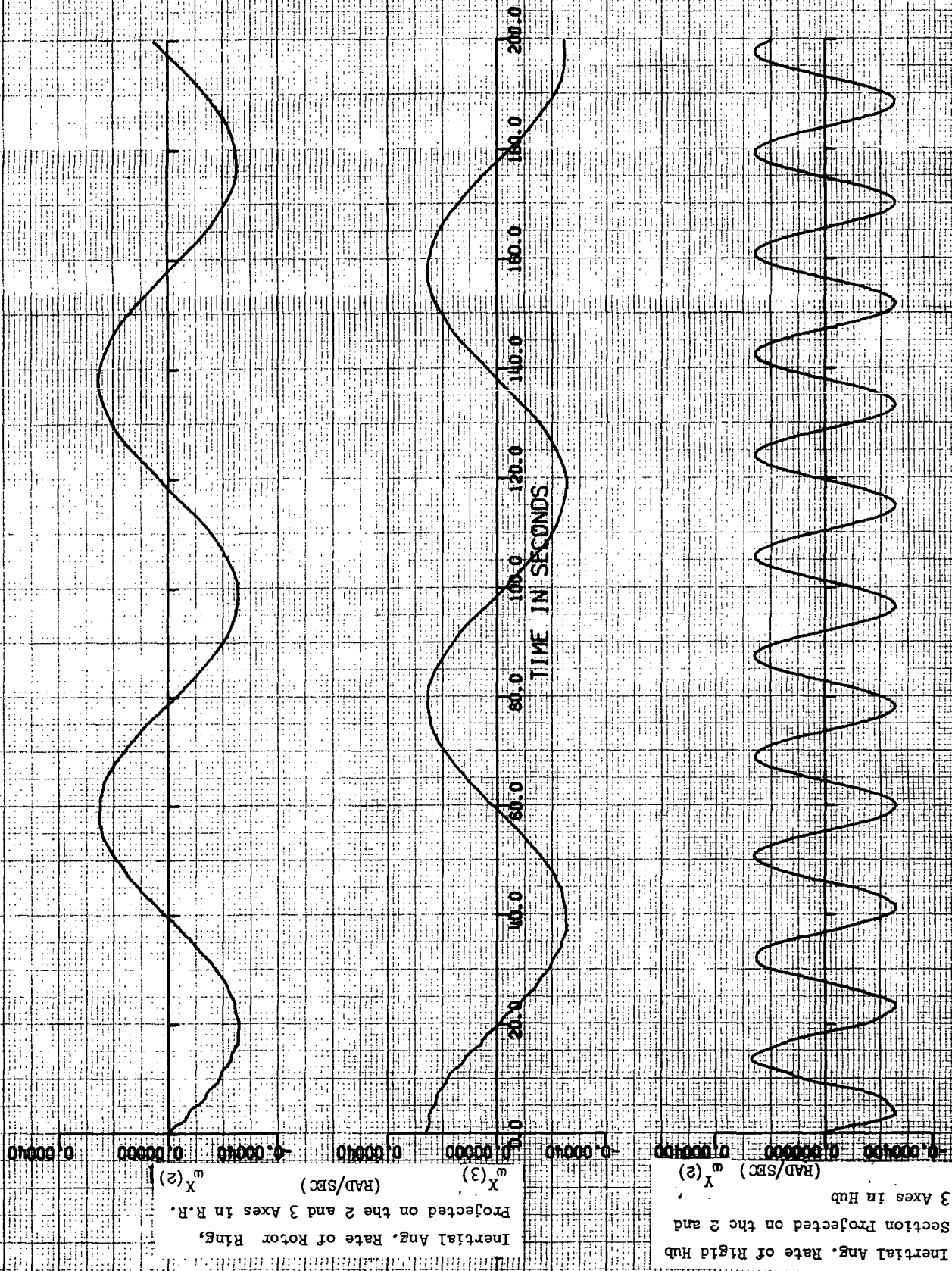


FIGURE 8.17 (Con't)

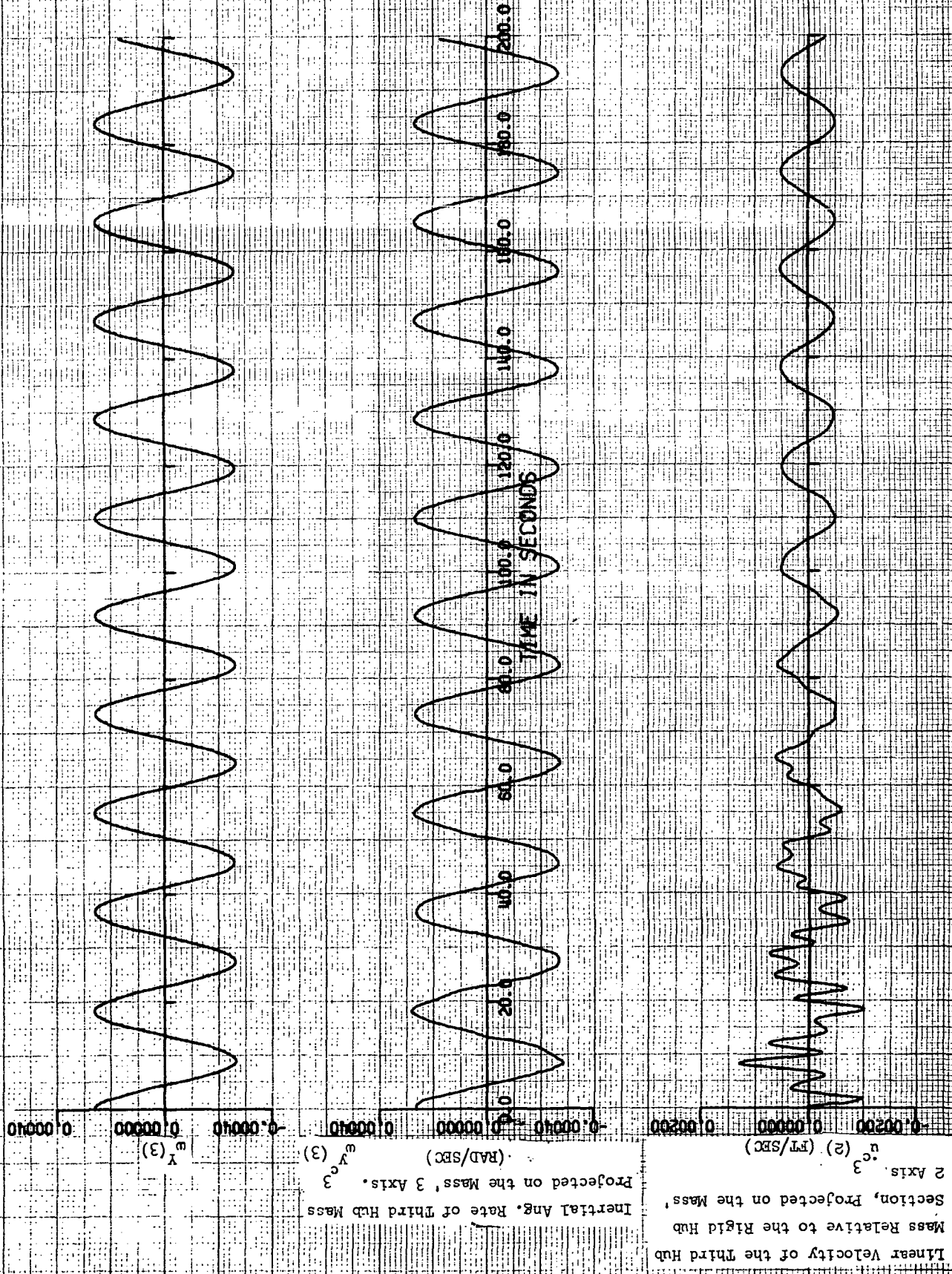
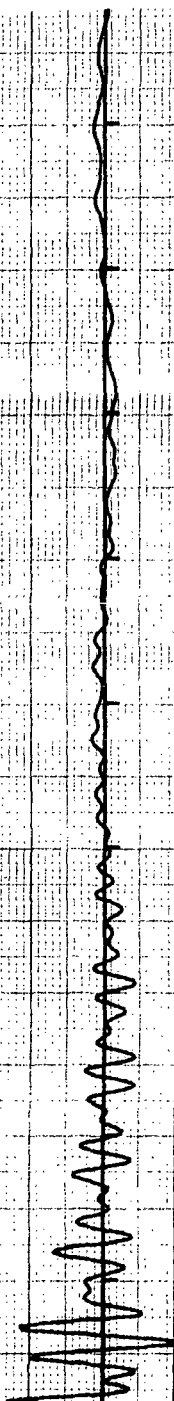


FIGURE 8.17 (con't)

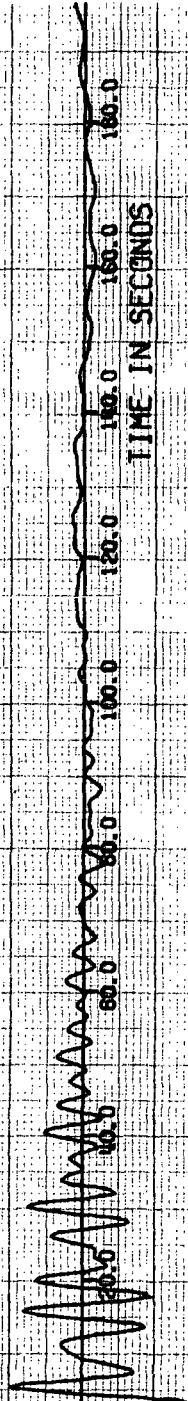
Linear Velocity of Third Rotor
Mass in the First Arm, Relative to
the Rotor Ring, Projected on the
Mass' 1 Axis.
 $v_{13}^{(1)}$ (FT/SEC)

0.0040
0.0000
-0.0040



Linear Velocity of the Third Rotor
Rotor Ring, Projected on the Mass' 1
and 2 Axes.
 $v_{23}^{(2)}$

0.0040
0.0000
-0.0040

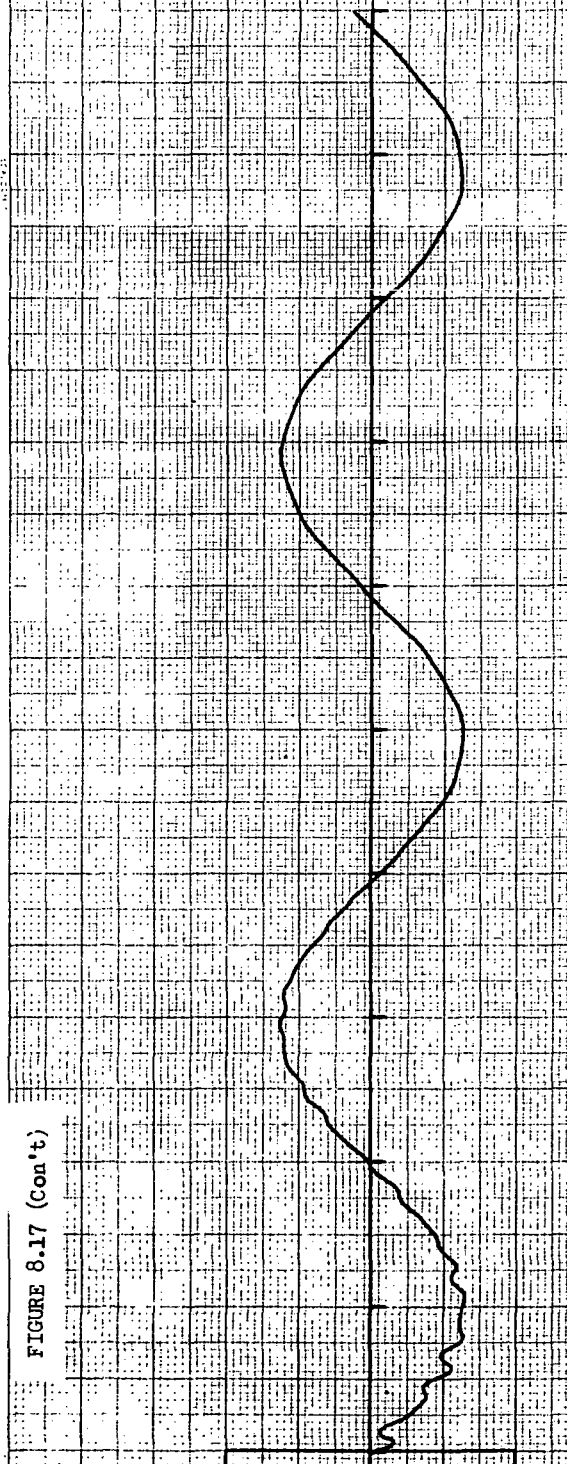


TIME IN SECONDS

0.0 20.0 40.0 60.0 80.0 100.0 120.0 140.0 160.0 180.0

FIGURE 8.17 (con't)

Inertial Angular Rate of the Third
 Rotor Mass in the First Arm, Projected
 on the Mass' 2 Axis.
 $\dot{\theta}_z$ (RAD/SEC)
 $\times 10^{-3}$



extension arms, both rotor and hub, increased the wobble damping. For example, the simulation condition with nominal stiffness experienced wobble decay with an estimated 2 hr. time constant in contrast to the 18.6 hr decay-time constant associated with perfectly stiff extension arms (as noted in Section 8.2, the results corresponding to bearing flexibility).

Comparing these time histories with those resulting from just a flexible bearing (Figure 8-9) it is seen that flexible motion has increased significantly as indicated by the high frequency oscillation, superimposed on the wobble motion. It should be noted that although a major portion of the flexible motion is damped during the course of the simulation run, there is a significant amount of flexible motion remaining which is being forced by the wobble motion (e.g., refer to the relative motion time histories of $\dot{\mu}^C$ and $\dot{\mu}^D$)

It is interesting to note that as the extension arm stiffness is altered the rigid body wobble characteristics change significantly. That is, as the extension arms were made stiffer it was observed that the wobble frequency increased. Specifically, the wobble period for different arm stiffness conditions are:

$$\text{"Y" Wobble Period} = \begin{cases} 93 \text{ sec, Baseline Stiffness} \\ 79 \text{ sec, Nominal Stiffness (2 x Base)} \\ 65 \text{ sec, Infinite Stiffness} \end{cases}$$

In other words, the effective transverse inertia of the total vehicle is lowered as the structure is made stiffer. A similar phenomenon was observed during studies when just the bearing stiffness was varied, and also during investigations of the flexible "T" configuration.

"Y" With Shuttle - Figure 8-18 are time histories resulting from the simulation of a "Y" space station with a shuttle flexibly attached to the rigid hub section. The initial conditions correspond to a nominal (or 1 sigma) docking disturbance as derived in Section 5.1. All of the physical properties are nominal (i.e., the arm stiffness is twice the baseline value) as presented in Section 4.0. As noted by the direction of rotation of the wobble vector, the addition of a shuttle transforms the "Y" configuration, which was

FIGURE 8.18 "Y"/SHUTTLE - NOMINAL, UNCONTROLLED

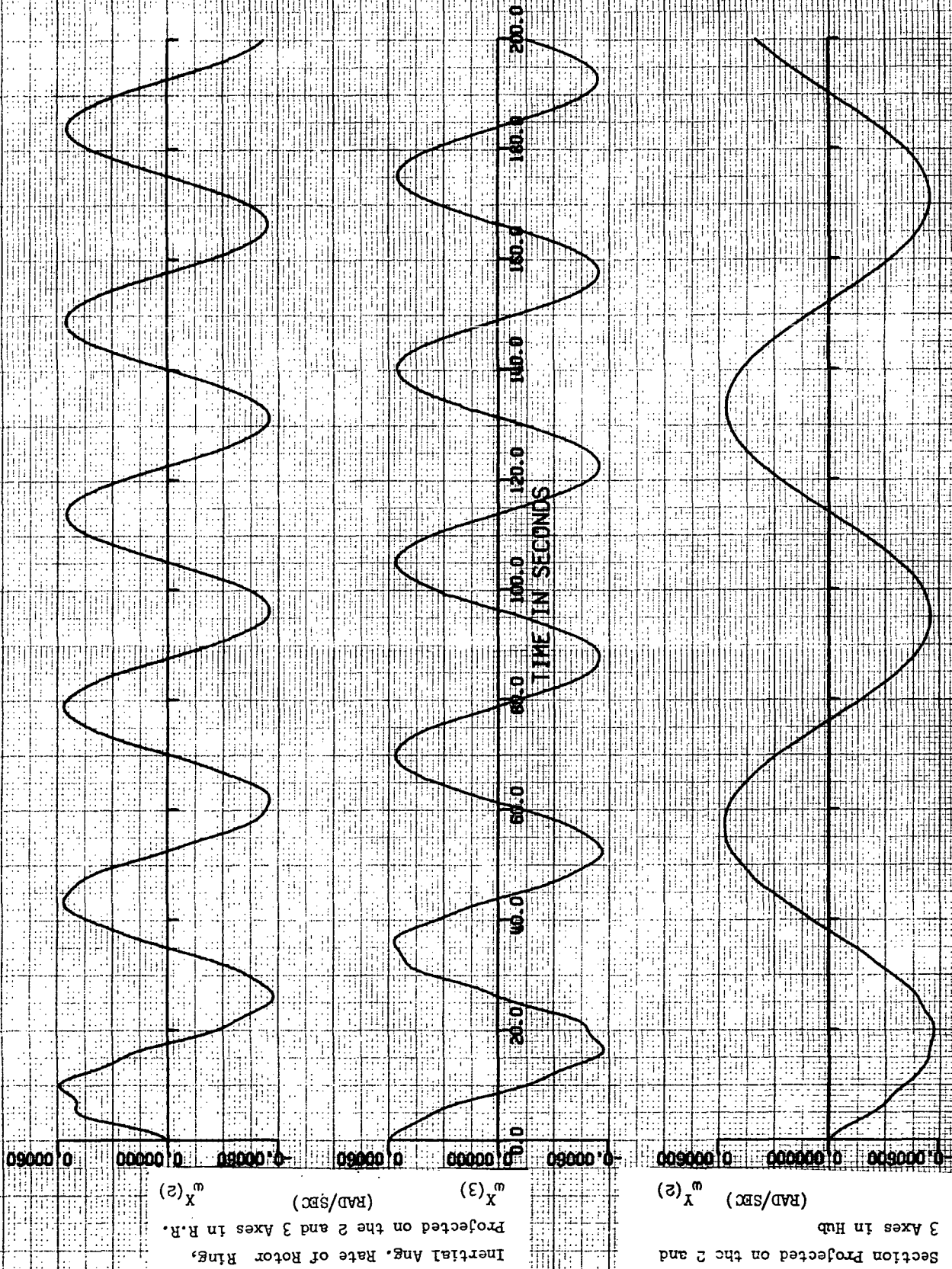


FIGURE 8.18 (Con't)

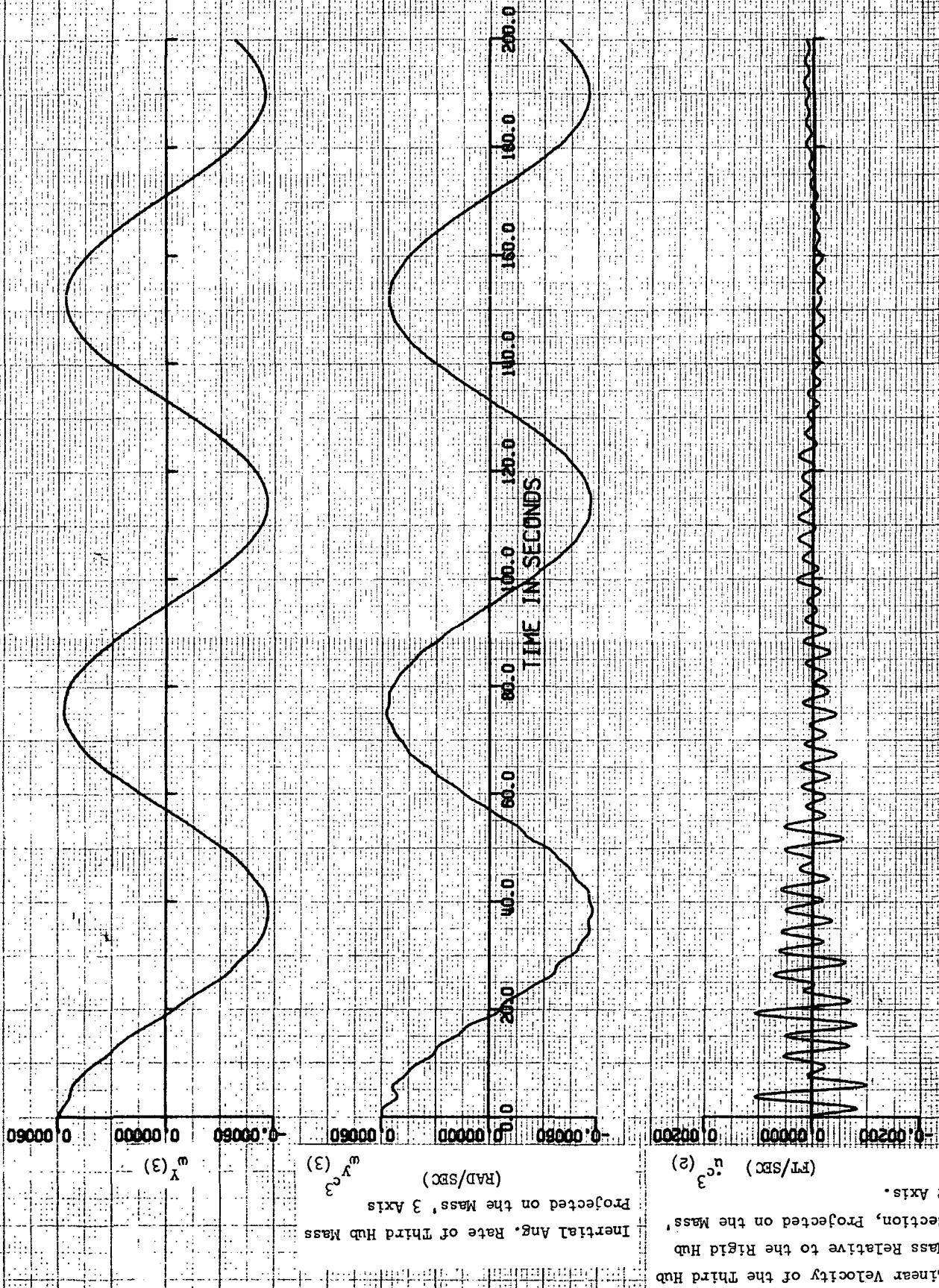


FIGURE 8.18 (Con't)

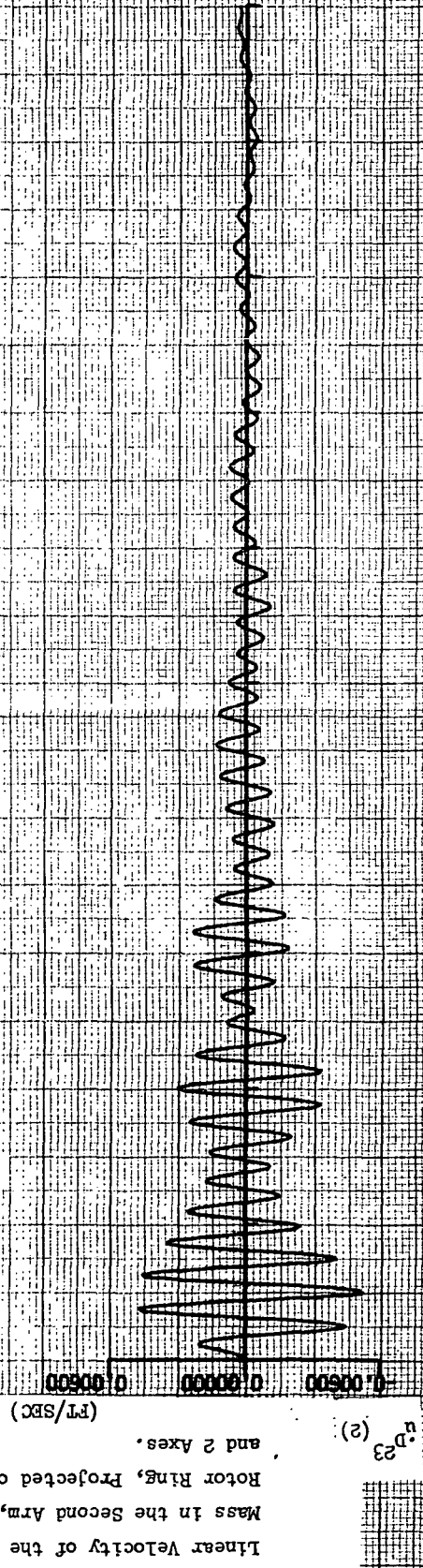
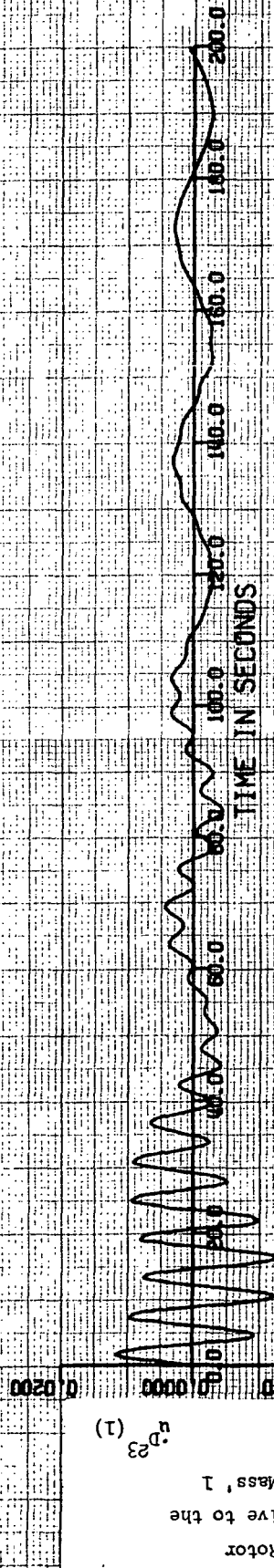
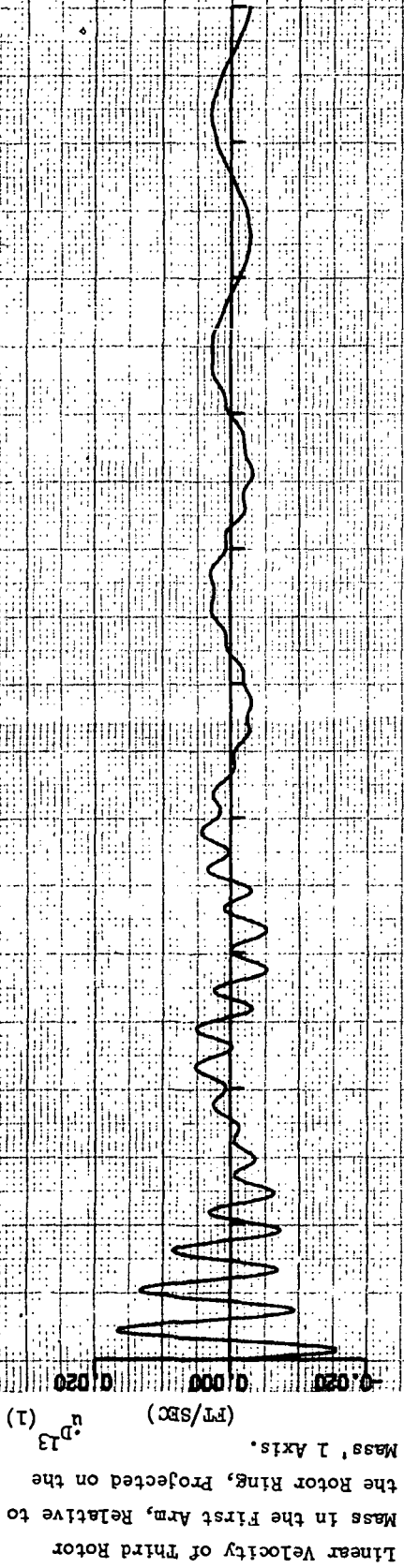
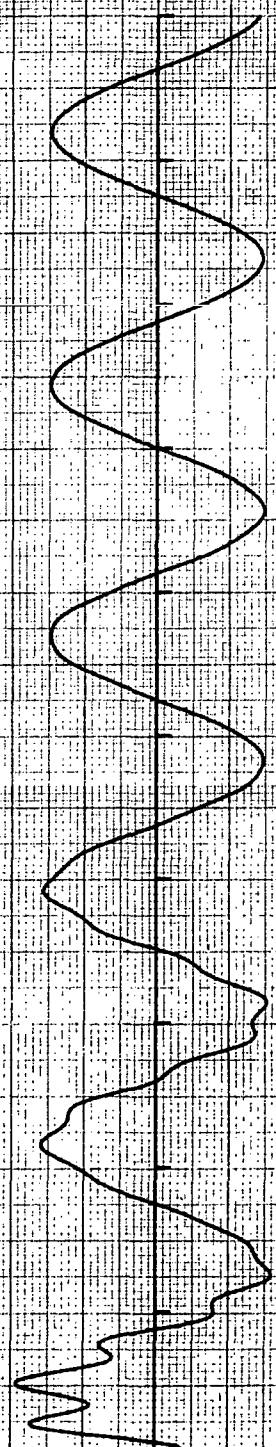


FIGURE 8.18 (Con't)

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis.

ω_{z13}^m (2)

-0.00080 0.00000 0.00080



originally "Max I", into a "Min I" configuration. Although this configuration was unstable with just bearing flexibility, the additional energy dissipative material that now is present in the hub and in the shuttle docking mechanism has stabilized the uncontrolled "Y"/Shuttle configuration. The estimated decay time constant, as obtained from printed simulation output data, is 0.73 hrs. This particular vehicle configuration is one of the few conditions investigated where the actual decay is observable on the time history traces.

T - Space Station - The time history traces shown in Figures 8-19 and 8-20 are the result of simulated "T" configuration tests with baseline and nominal stiffness conditions, respectively. The initial conditions represent a disturbance resulting from internal mass motion (refer to Section 5.2). Although all the structural stiffness conditions investigated resulted in a seemingly stable wobble mode, the stability is extremely marginal. That is, the decay time constant is of the order of several hours. In addition, as the wobble mode approaches neutral stability the accuracy to which a time history program can detect absolute stability is questionable. If a more accurate determination of stability is required then a characteristic root analysis should be performed, which would require the application of Floquet Theory to accommodate the time varying coefficients (refer to Reference 9).

A simulated condition of a "T" station with infinitely stiff hub in combination with nominal flexible rotor arms and bearing was investigated. The resulting time histories are shown in Figure 8-21. The wobble mode for this condition is definitely unstable. Therefore, energy dissipative material added to the hub has a stabilizing influence as indicated by the previously described results where marginal stability was observed.

When viewing the time history traces it is observed that the amplitude of the relative flexible motions are generally higher for the "T" configuration as opposed to "Y" configuration results with identical stiffness characteristics. These larger flexible motions are attributed to the mass distribution associated with the "T" configuration.

As experienced during "Y" investigations, the frequency of the rigid body wobble mode varied with changes in structural stiffness. Consistent with the "Y" results, an increase in arm stiffness caused a decrease in the effective total

FIGURE 8.19 "Y" - BASELINE, UNCONTROLLED

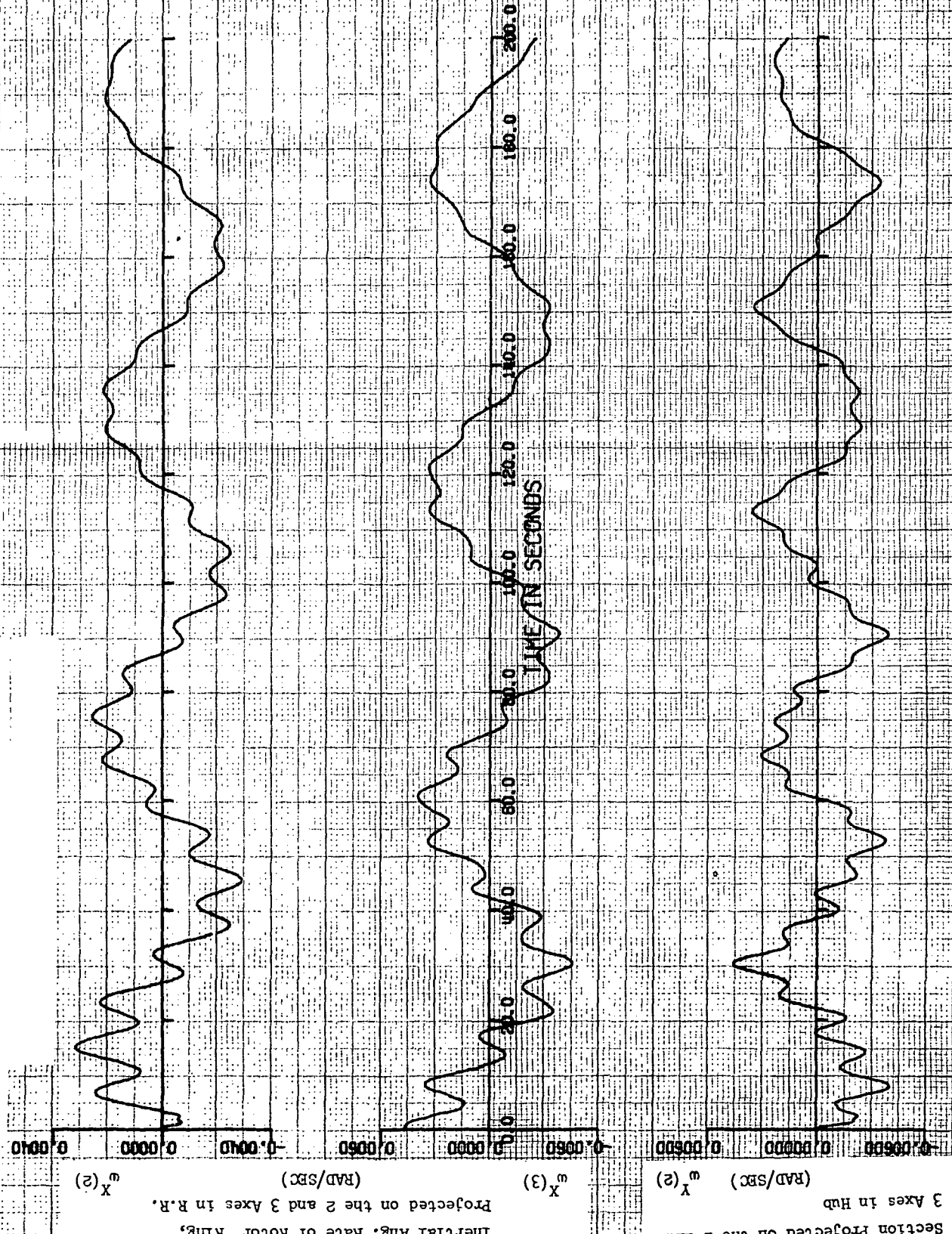
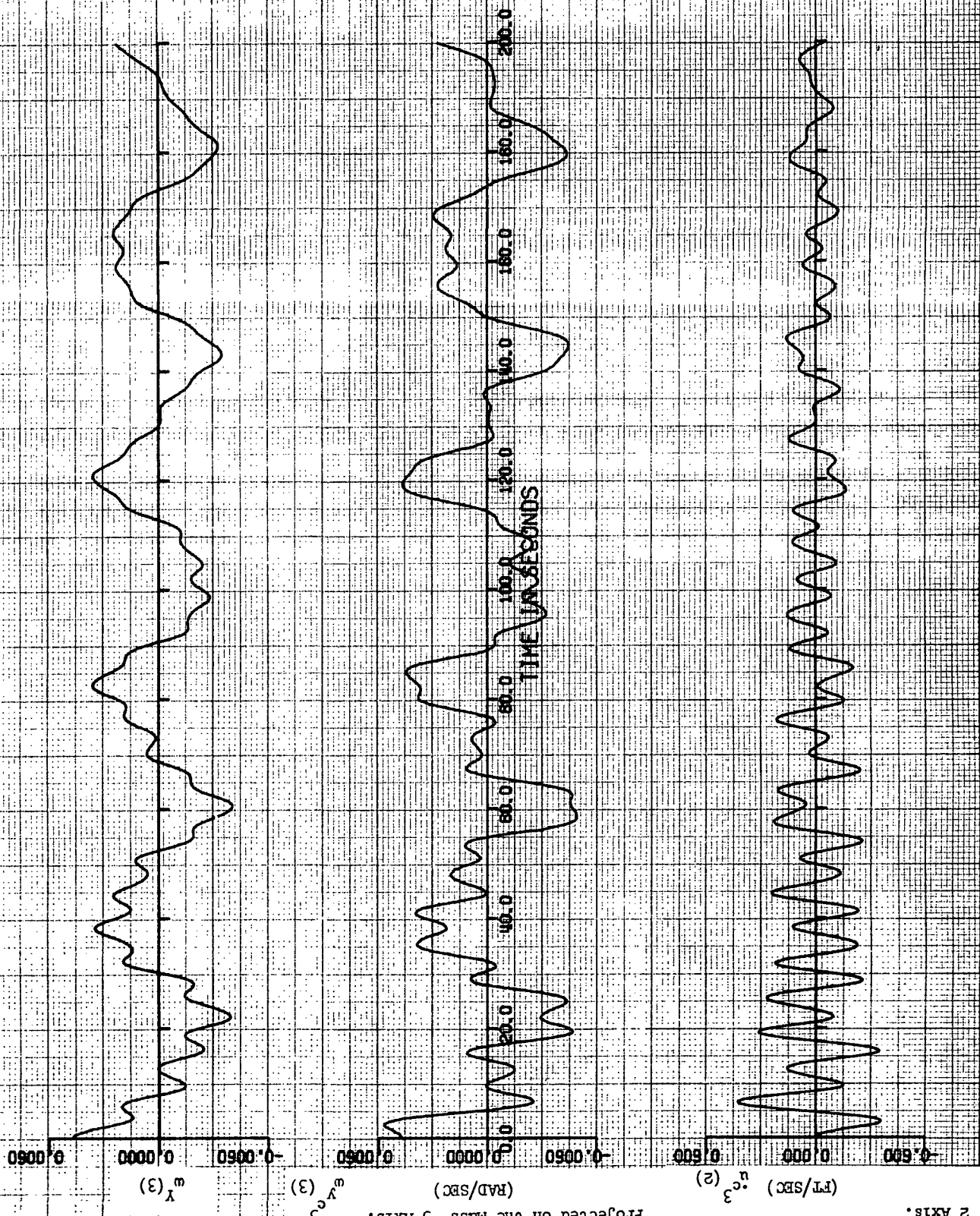


FIGURE 8.19 (con't)



Linear Velocity of the Third Hub
 Mass Relative to the Rigid Hub
 Section, Projected on the Mass'
 2 Axis.

Inertial Ang. Rate of Third Hub Mass
 Projected on the Mass' 3 Axis.

\ddot{u}_3^c (3)

\dot{u}_3^c (3)

\dot{u}_3^c (3)

\dot{u}_3^c (2)

0.0050

0.0000

-0.0050

0.0050

0.0000

-0.0050

0.600

0.000

-0.600

TIME IN SECONDS

200.0

180.0

160.0

140.0

120.0

100.0

80.0

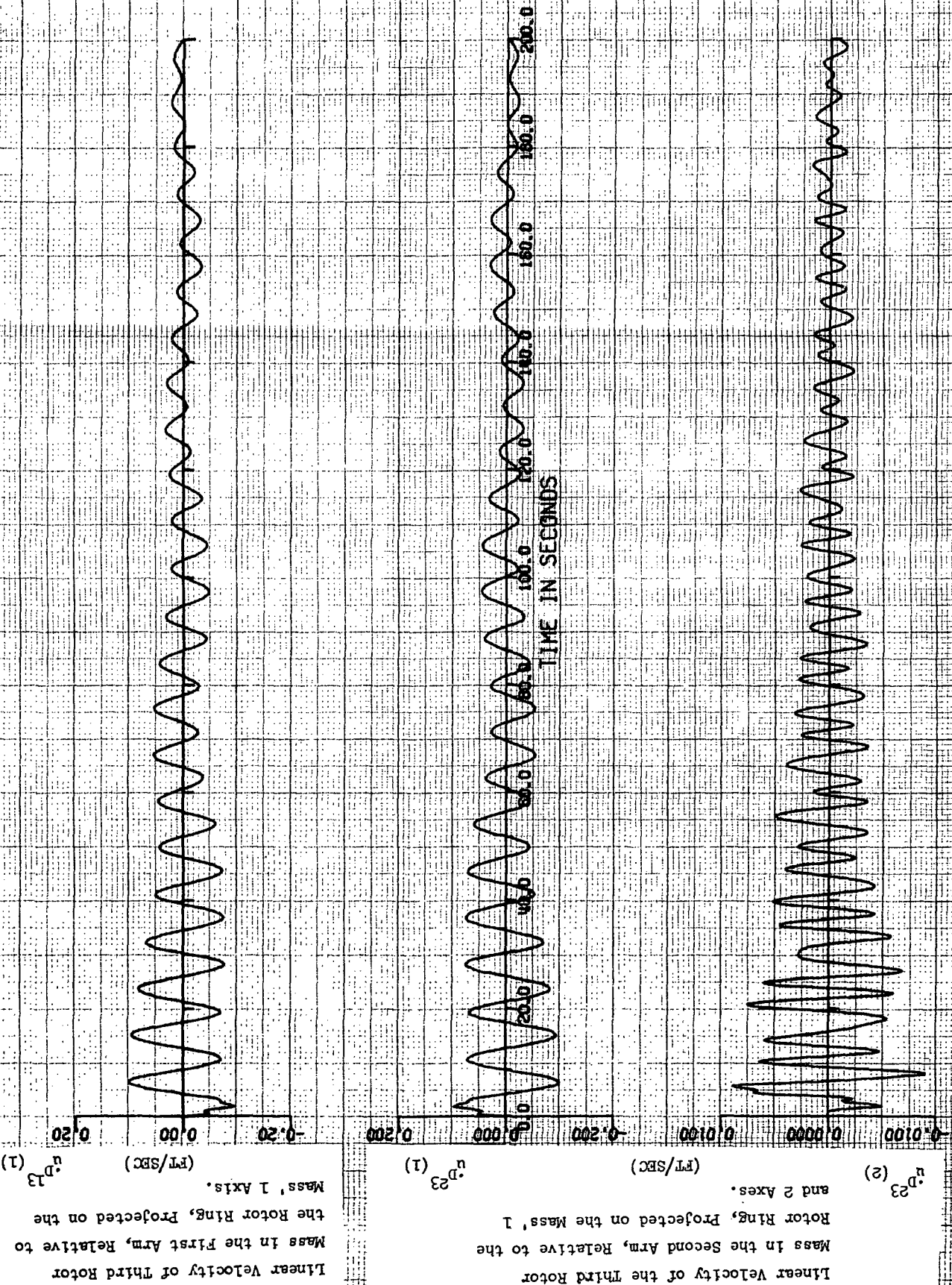
60.0

40.0

20.0

0.0

FIGURE 8.19 (con't)

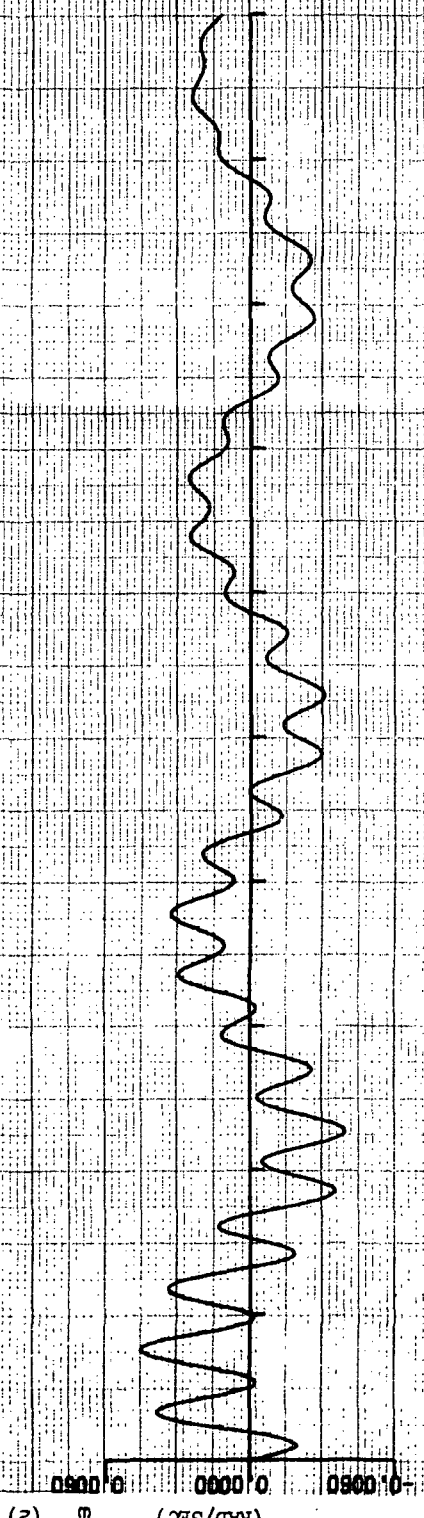


Linear Velocity of Third Rotor
Mass in the First Arm, Relative to
the Rotor Ring, Projected on the
Mass' 1 Axis.

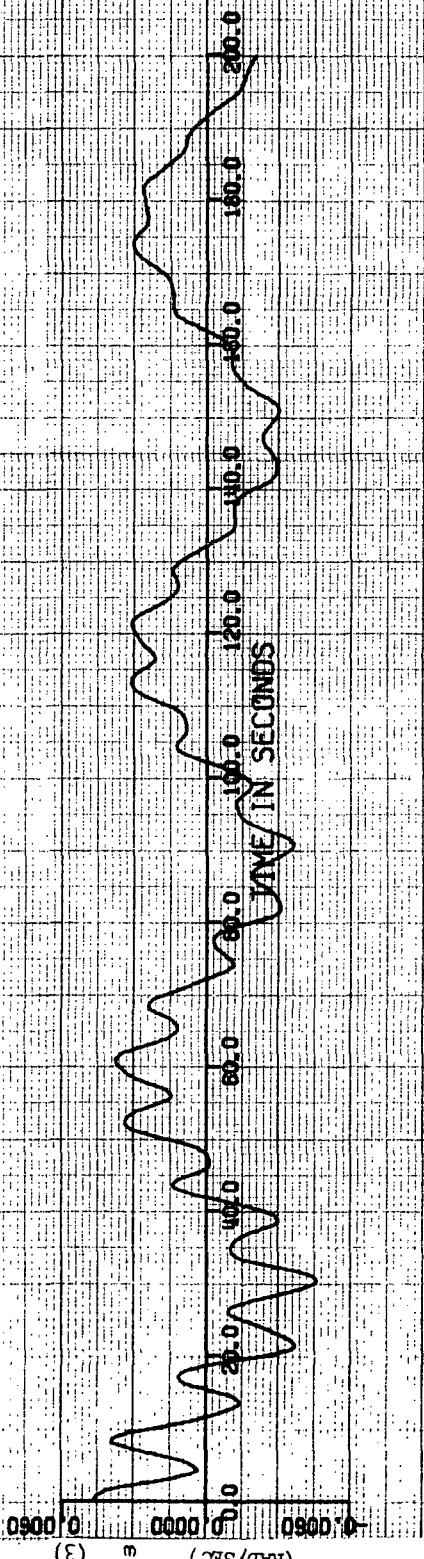
Linear Velocity of the Third Rotor
Rotor Ring, Projected on the Mass' 1
and 2 Axes.

FIGURE 8.19 (Con't)

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis.



Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 3 Axis



Inertial Angular Rate of the Third Rotor Mass in the Second Arm, Projected on the Mass' 2 Axis

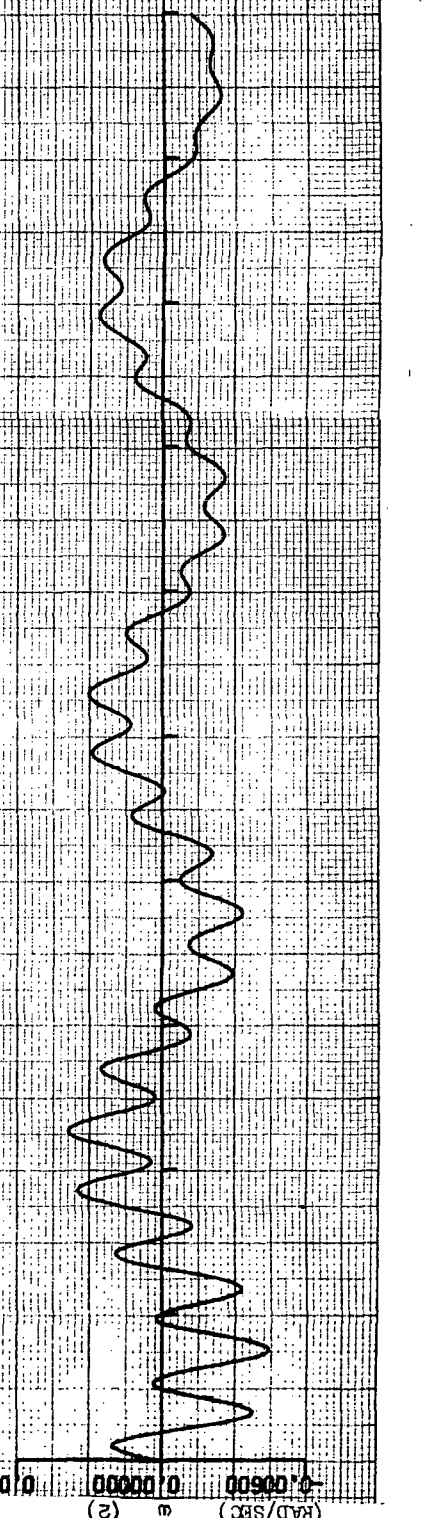


FIGURE 8.20 ω_{in} - NOMINAL, UNCONTROLLED

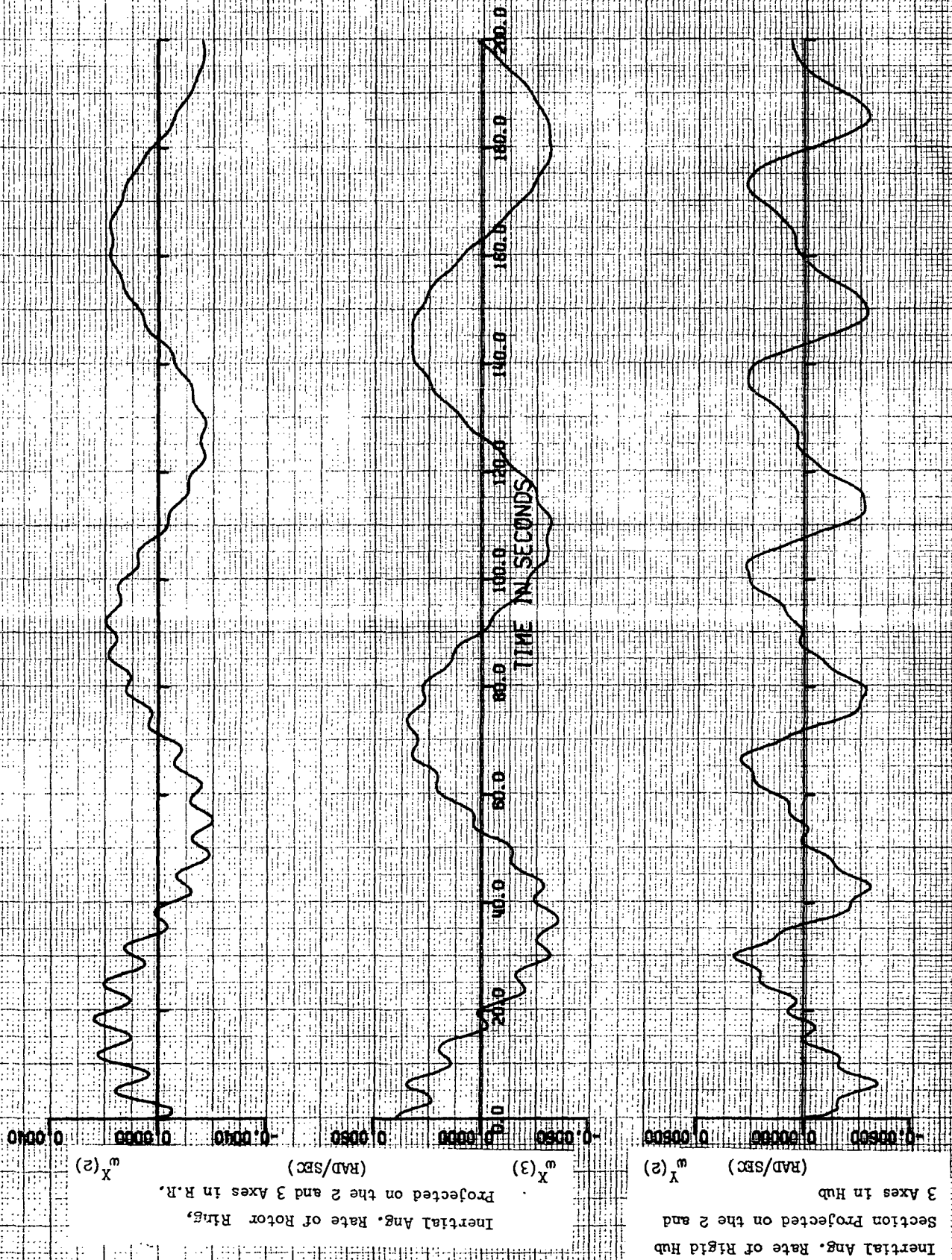


FIGURE 8.20 (Con't)

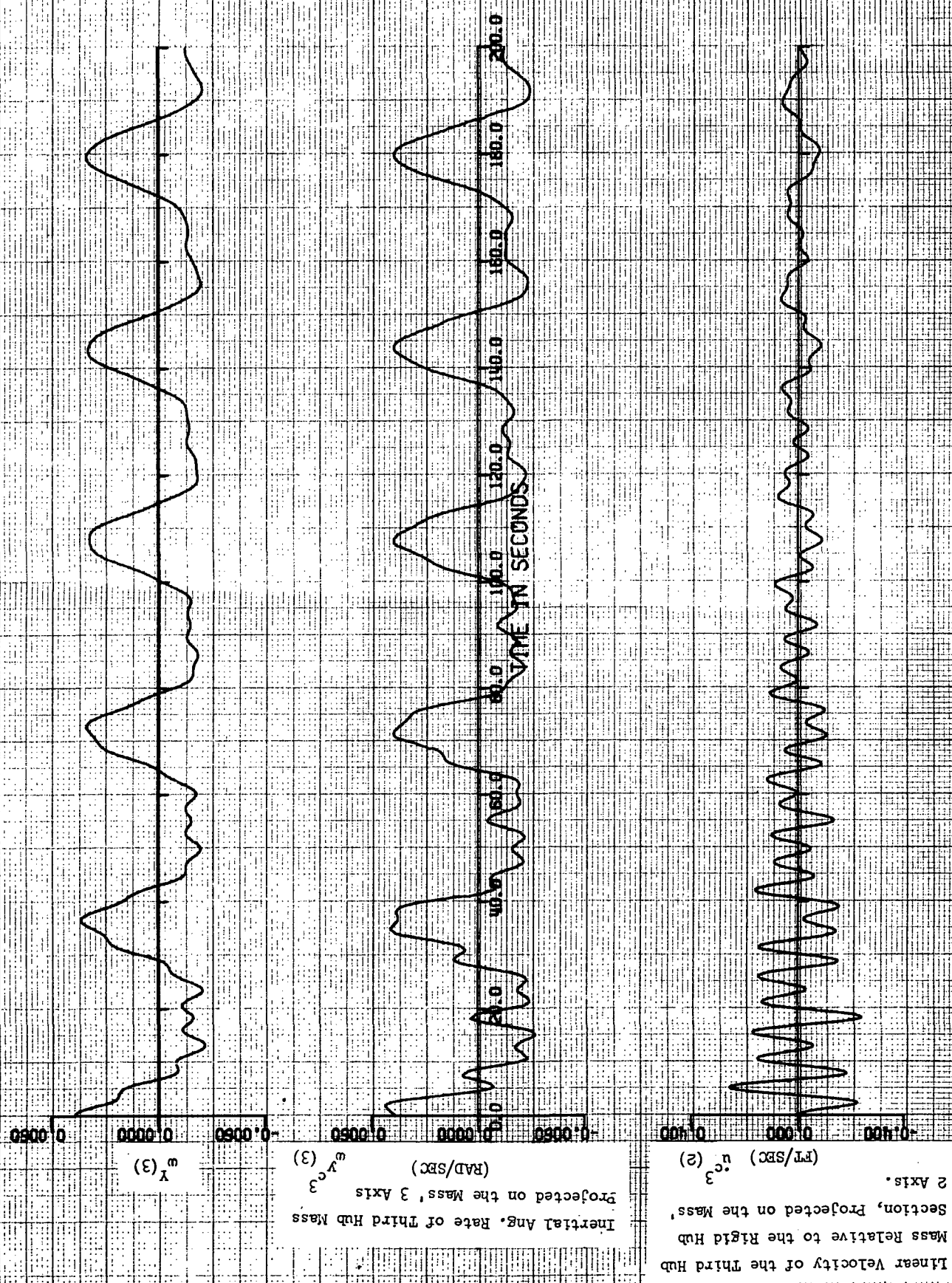


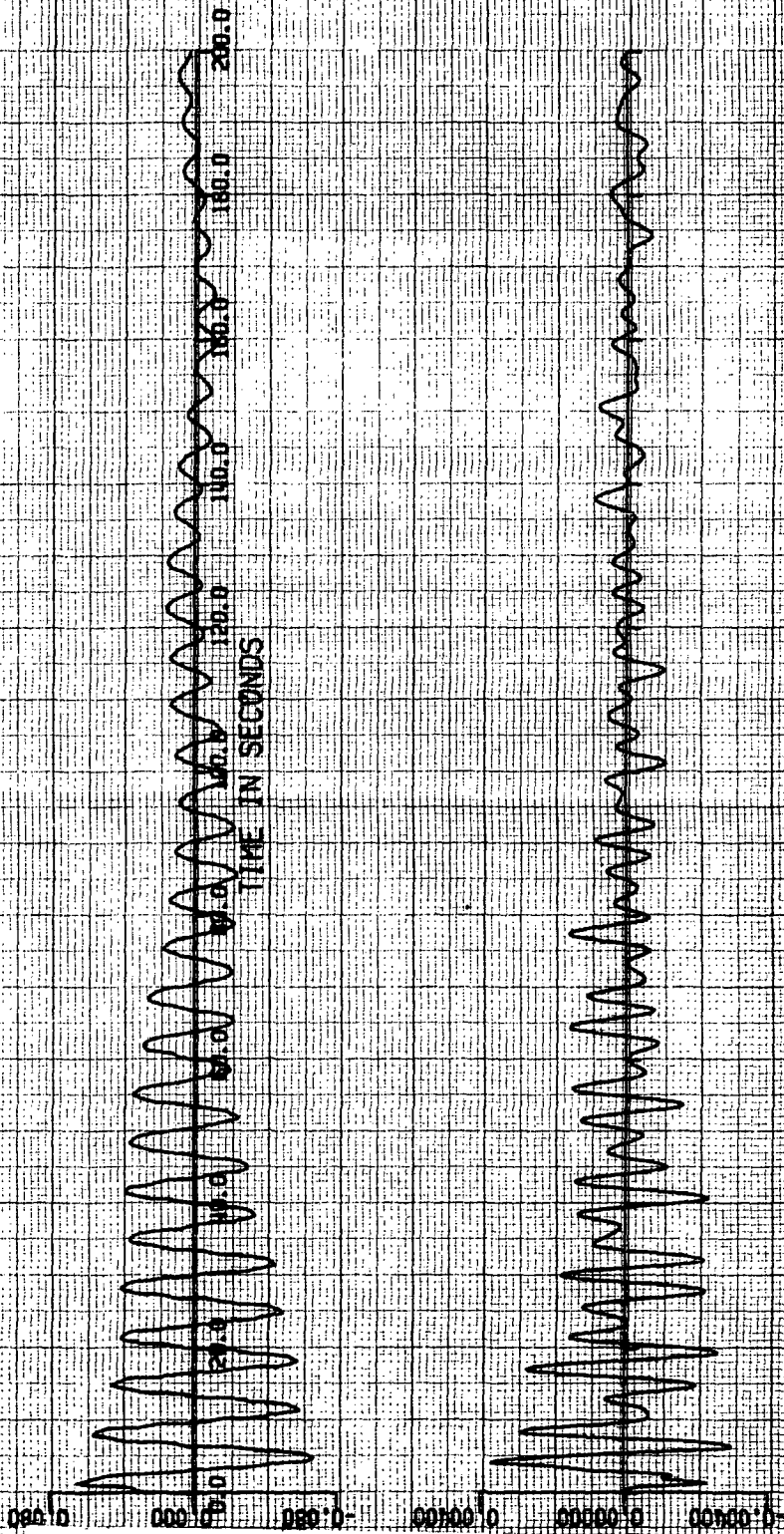
FIGURE 8.20 (Con't)

Shuttle Ang. Rate Relative to
the Hub, Projected on Shuttle
1 Axis.
(RAD/SEC) $\psi(1)$

Linear Velocity of the Third Rotor
Mass in the Second Arm, Relative to the
Rotor Ring, Projected on the Mass 1
and 2 Axes.
(FT/SEC) $v_{23}^{(2)}$

$v_{23}^{(1)}$

$v_{23}^{(2)}$



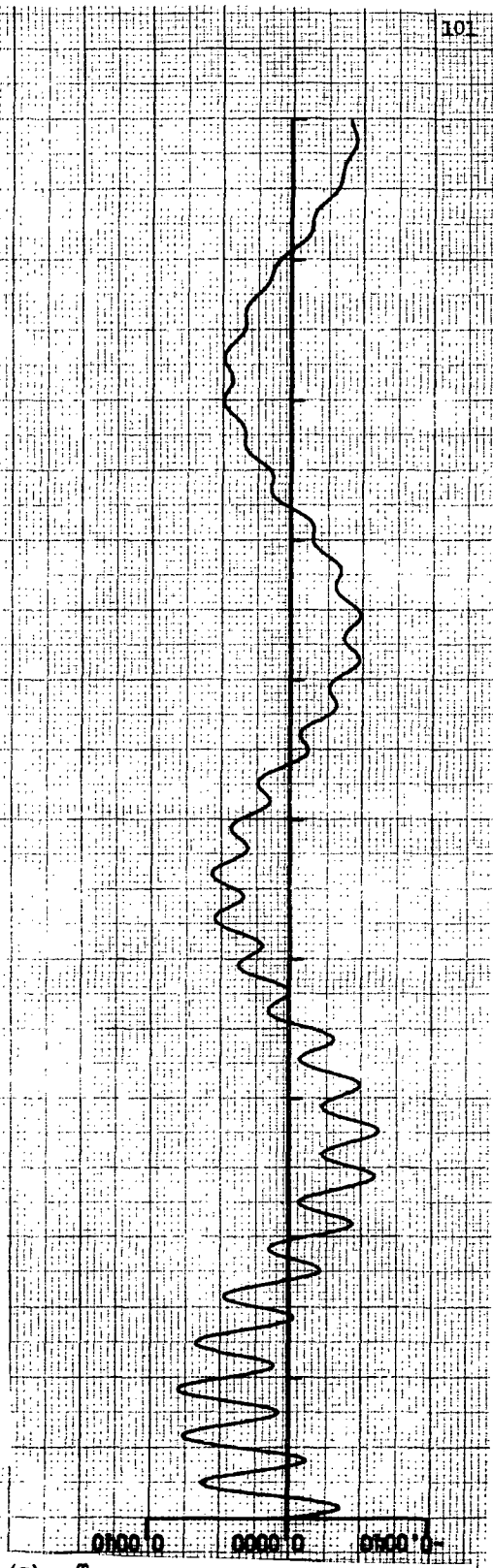


FIGURE 8.20 (Con't)

Inertial Angular Rate of the Third
Rotor Mass in the First Arm, Projected
on the Mass' 2 Axis.
(RAD/SEC) z_{13} (2)

FIGURE 8.21 $\dot{\theta}_T$ WITH RIGID HUB - NOMINAL ROTOR, UNCONTROLLED

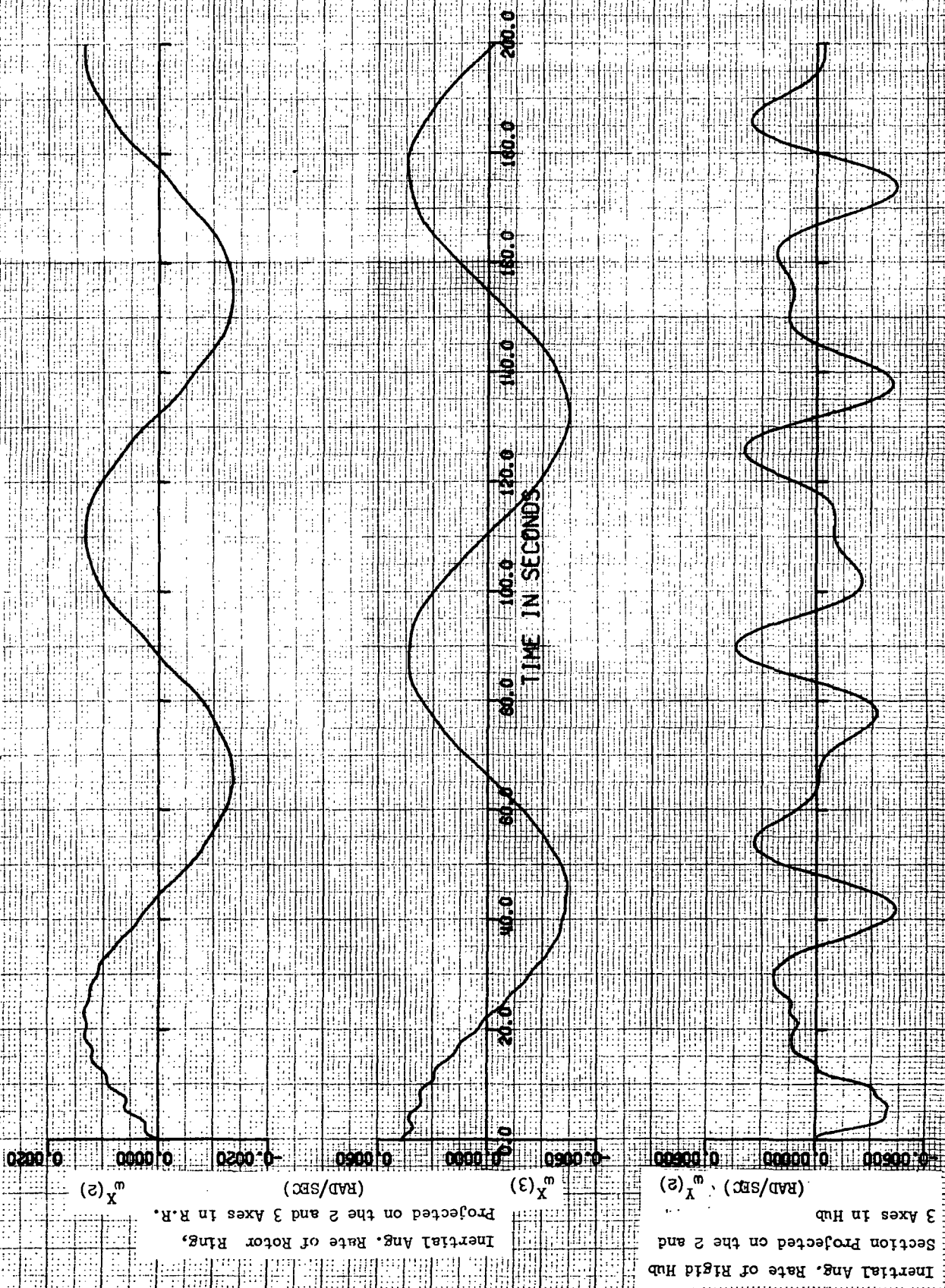


FIGURE 8.21 (Con't)

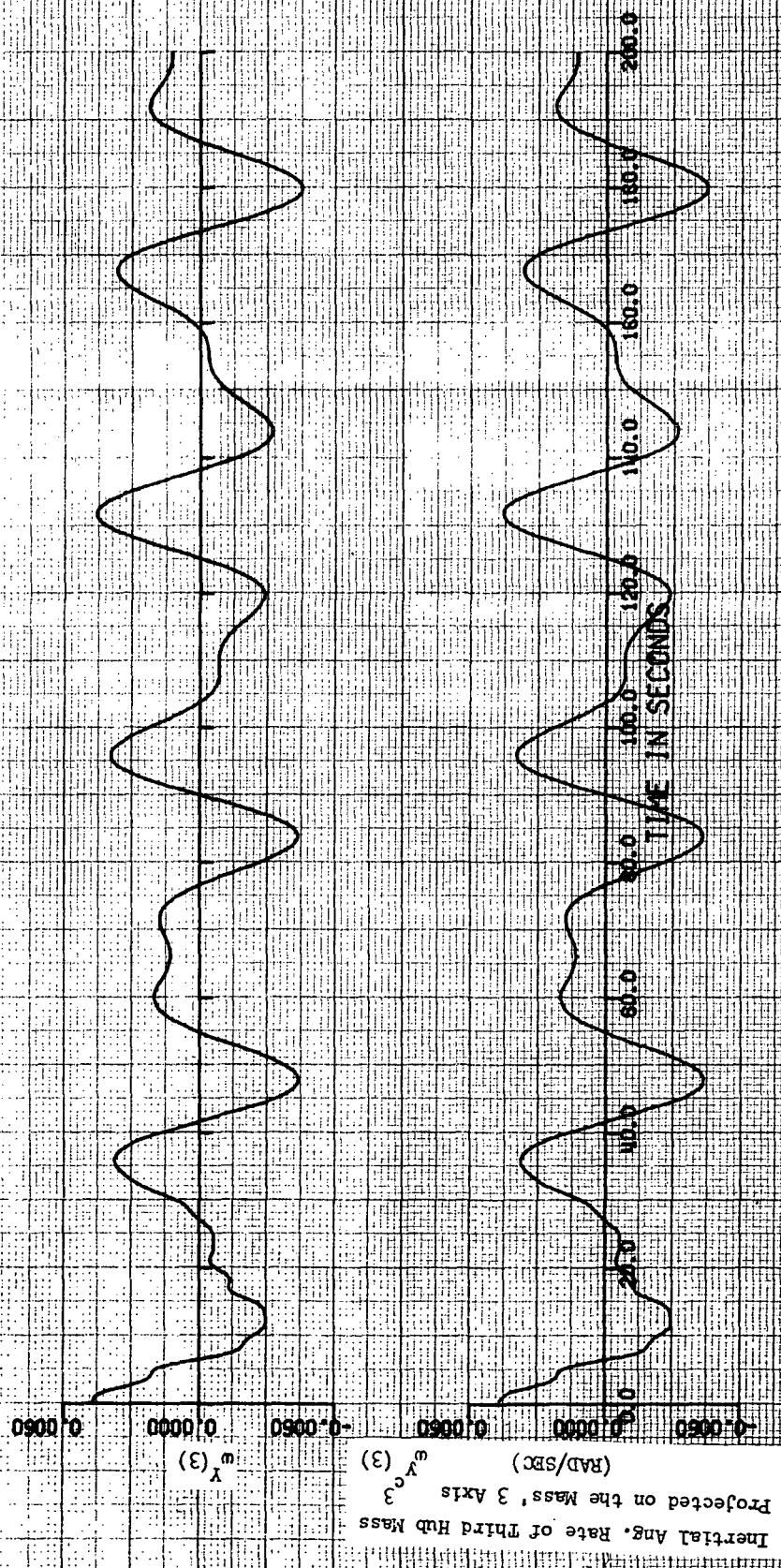
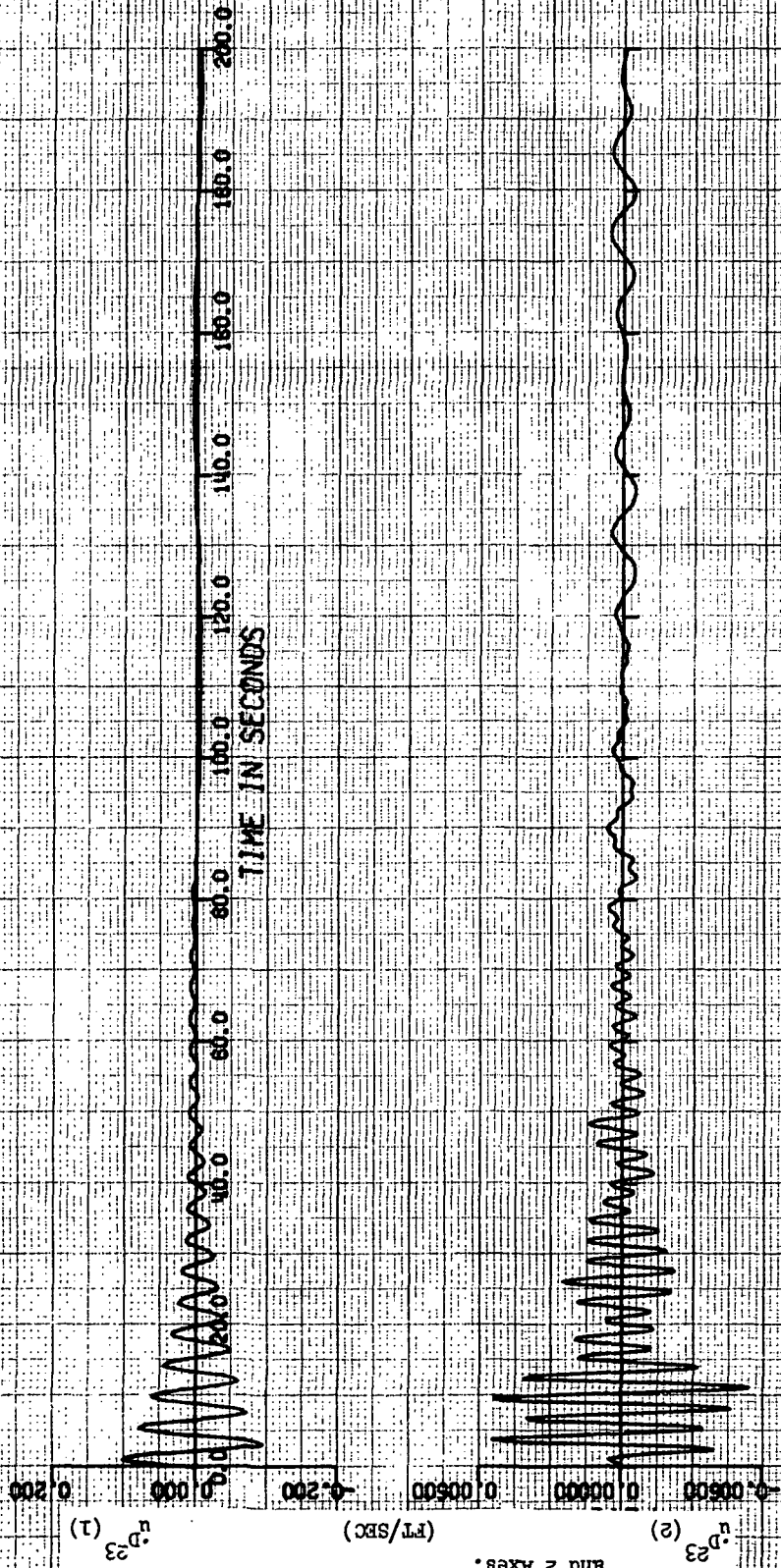


FIGURE 8.21 (Con't)

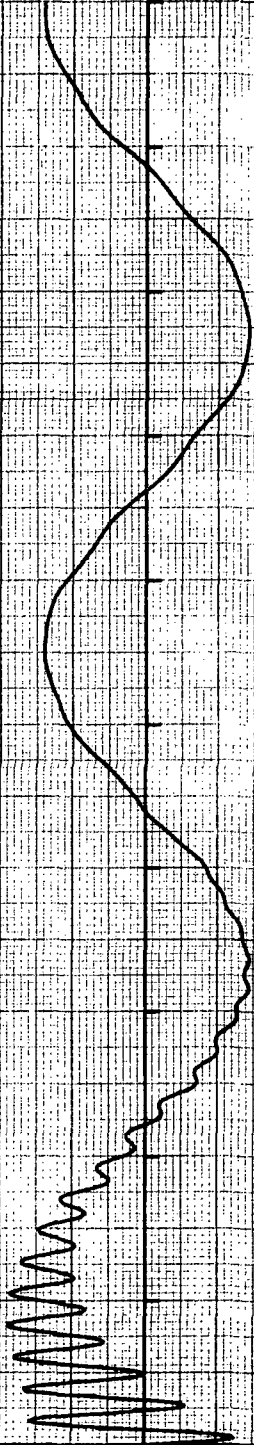


Linear Velocity of the Third Rotor
Rotor Ring, Projected on the Mass 1
and 2 Axes.

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis. ω_{z13}^2 (RAD/SEC)

0.0020 0.0000 -0.0020

FIGURE 8.21 (Con't)



vehicle transverse inertia. However, in the case of the "T" configuration a decrease in transverse inertia results in a decrease in wobble frequency. Specifically, the wobble period for different arm stiffness conditions are:

$$T \text{ Wobble Period} = \begin{cases} 59 \text{ sec, Baseline Stiffness} \\ 73 \text{ sec, Nominal Stiffness} \\ 91 \text{ sec, Infinite Stiffness} \end{cases}$$

"T" With Shuttle - The time histories resulting from the simulation of the "T" space station with a flexibly attached shuttle are shown in Figure 8-22. All of the physical properties are nominal as presented in Section 4.0. The initial conditions correspond to a 1 sigma docking disturbance (refer to Section 5.1). The wobble mode of this configuration is also marginally stable with a decay time constant of the order of a few hours. Again, there is sufficient energy dissipation in the shuttle docking mechanism and in the despun hub to stabilize this uncontrolled "Min I" configuration. An additional time history trace was added to this set of traces in order to indicate the extent of the relative motion between the shuttle and the station. Examination of ψ indicates damped relative motion with a residual wobble forced vibration.

It is interesting to note that this configuration is no longer close to being an "Intermediate Inertia" vehicle. As a result, a predominate frequency equal to spin minus wobble frequency ($p - \lambda p$) is associated with the wobble motion as viewed in the hub.

FIGURE 8.22 "T" SHUTTLE - NOMINAL, UNCONTROLLED

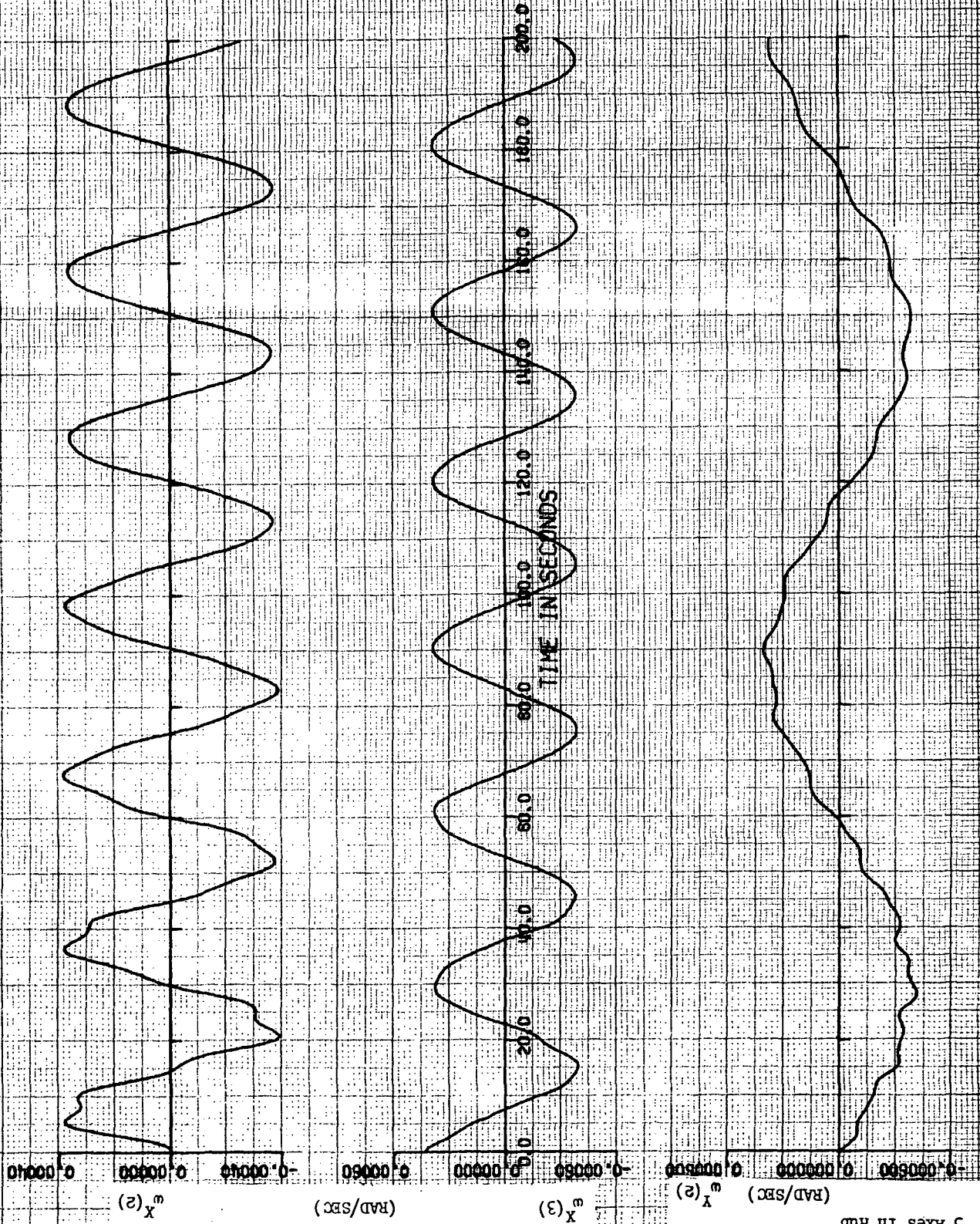


FIGURE 8.22 (con't)

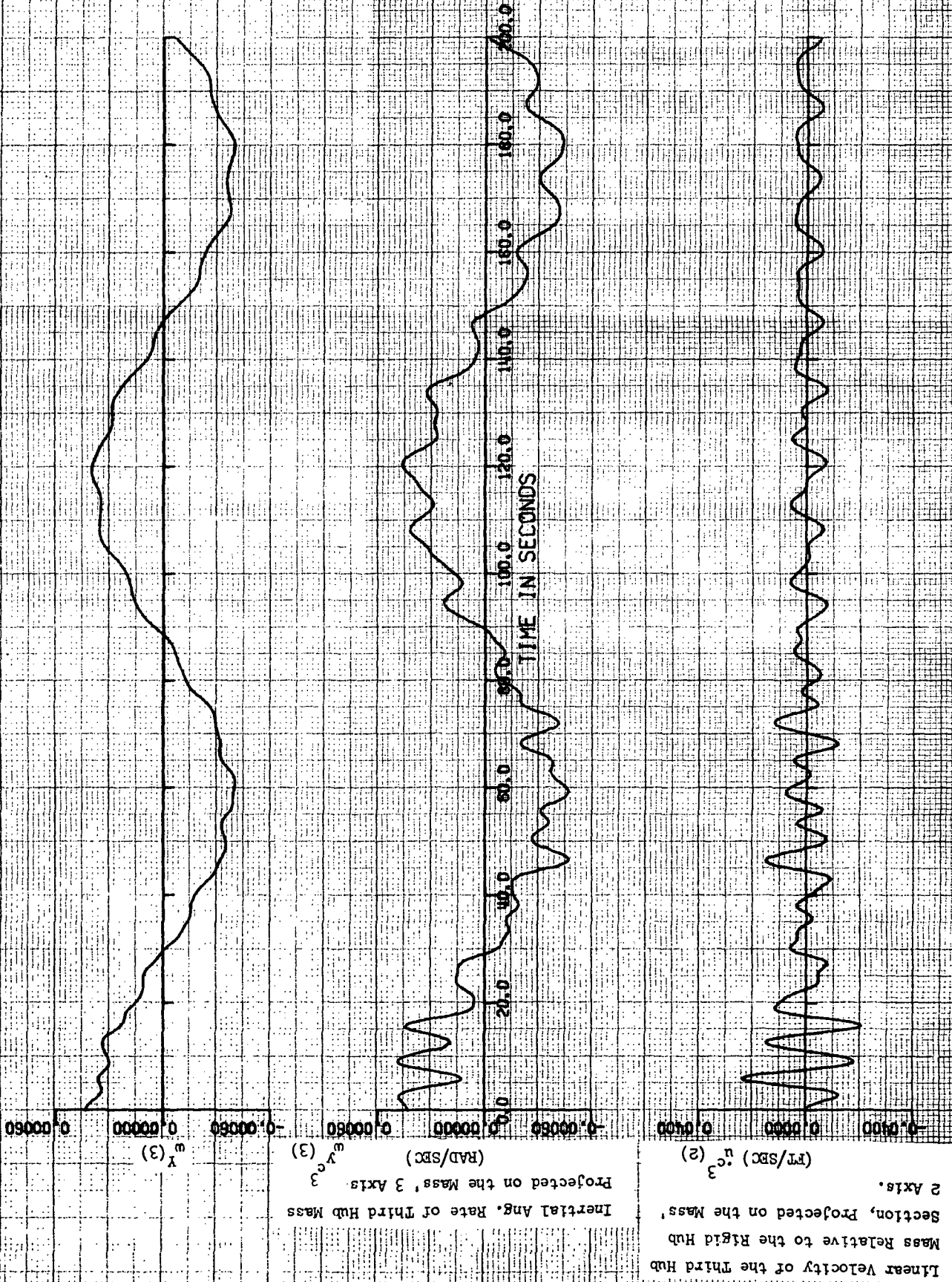
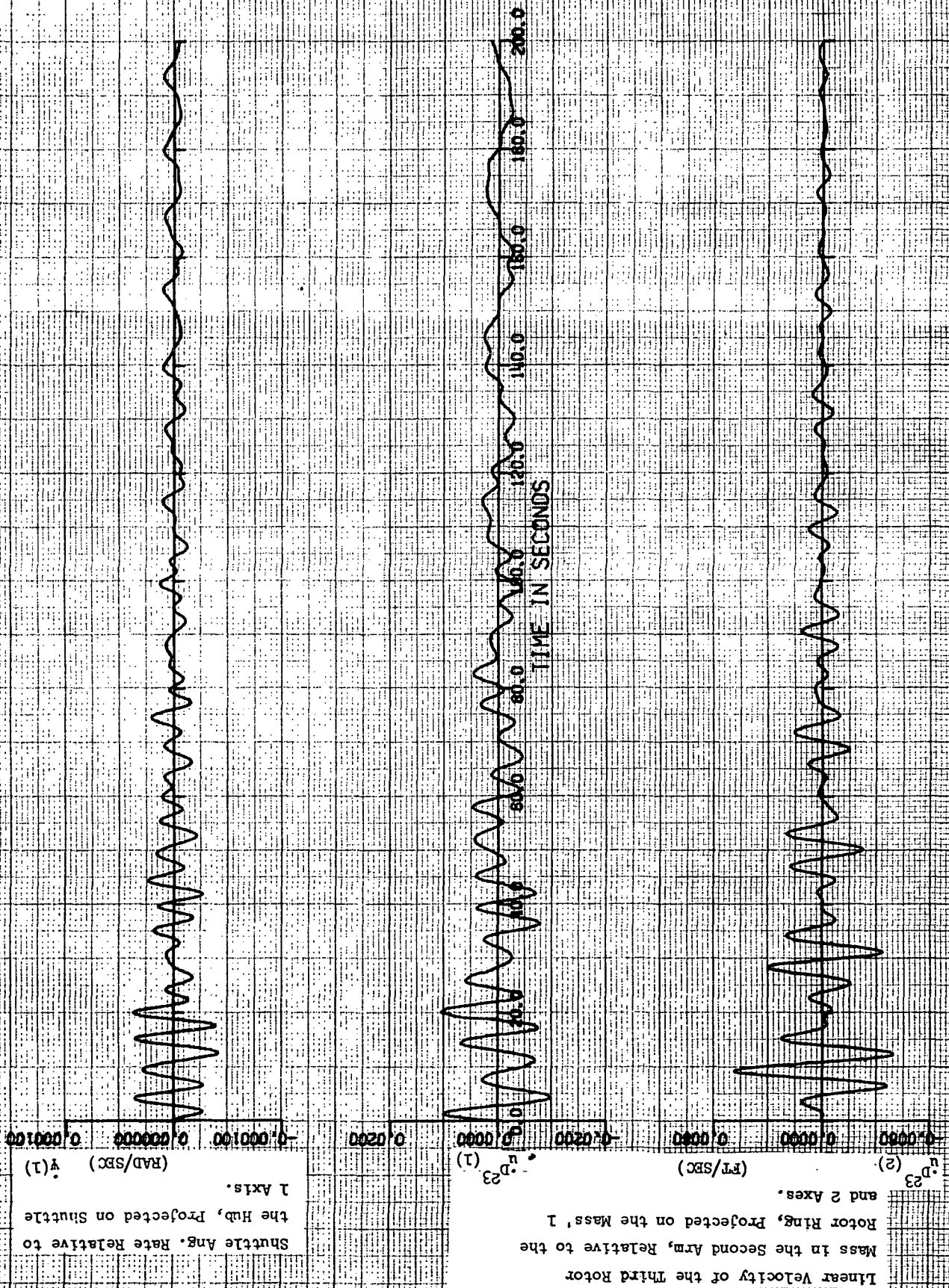


FIGURE 8.22 (con't)



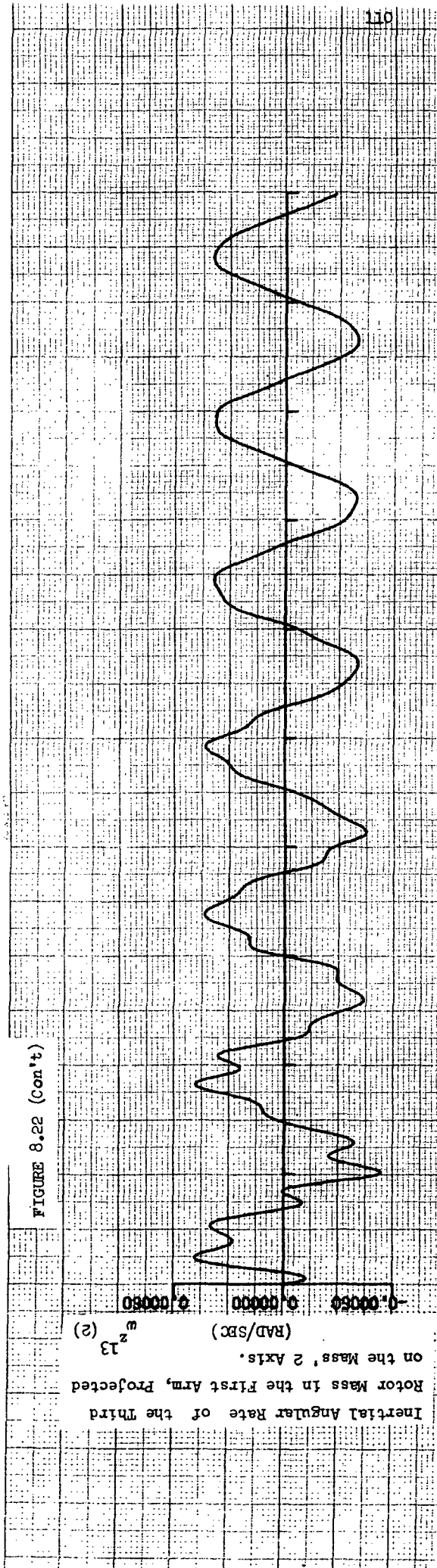


FIGURE 8.22 (con't)

Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis.

(RAD/SEC) $m^2 13$ (2)

-0.00050 0.00000 0.00050

9.0 CMG Wobble Damping

9.1 Preliminaries

The formulation of the preferred CMG wobble damping technique evolved from a series of analyses beginning with the previous study Grumman performed for NASA/MSC concerning CMG applications to the Space Base (Reference 3). After investigating many different CMG configurations and associated control laws, a unique wobble damping concept was developed. The CMG control technique employs a single-gimbal CMG with the gimbal axis aligned to the vehicle spin axis. The associated control strategy, which is illustrated in Figure 9-1, is to drive the gimbal so that the CMG spin axis (i.e., the CMG momentum vector, h) lags the wobble vector $\{\omega_T\}$ by 90 degrees in inertial space. This strategy is called the 90 degree H-Lag Law. The above control technique was chosen because it accomplishes the following desirable control characteristics:

- o The transverse component of the control torque is always in opposition to the wobble vector.
- o The total torque impulse (or angular momentum) applied in opposition to the wobble, per wobble period, is near the maximum deliverable as limited by the wheel size, h .

The latter characteristics can be demonstrated as follows:

The control torque $\{T_c\}$ is given by,

$$\{T_c\} = - \{\dot{h}\} = - \{\omega_{CMG}\} \times \{h\} \quad (9-1)$$

where

$\{\dot{h}\}$ = the inertial time derivative of the CMG angular momentum vector.

$$\begin{aligned} \{\omega_{CMG}\} &= \text{total inertial angular rate of the CMG} \\ &= \{\omega\} + \{\omega_G\} \end{aligned}$$

$$\{\omega\} = \text{total vehicle angular rate} = P \{i\} + \{\omega_T\}$$

$$\{\omega_G\} = \text{angular rate of the CMG gimbal relative to the vehicle}$$

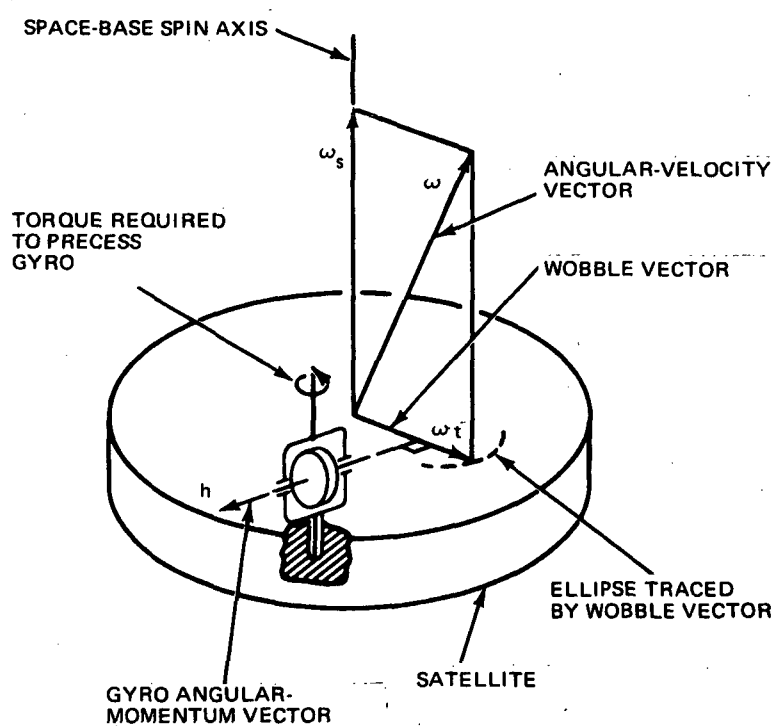


Figure 9-1. CMG 90-deg H-Lag Law

and the transverse torque component $\{T_{C_T}\}$ is,

$$\{T_{C_T}\} = - (P + \omega_G) \{i\} \times \{h\} \quad (9-2)$$

If $\{h\}$ lags $\{\omega_T\}$ by 90 degrees then it is easily seen that the above torque component will always oppose $\{\omega_T\}$, provided $\omega_G > -P$. Furthermore, in order to track the wobble vector and maintain the proper 90 degree phase relationship, the gimbal rate, ω_G , is constrained to equal the rotation rate of the wobble vector, which will always be greater than minus spin. The wobble vector rotation rate $\dot{\phi}$ can be derived by defining ϕ as,

$$\tan \phi = \frac{\omega_Y}{\omega_Z} \quad (9-3)$$

Using equation (8-1) for the wobble components ω_Y and ω_Z , differentiating the result and making the appropriate trigonometric substitutions results in,

$$\dot{\phi} = \pm \lambda P \frac{2\alpha}{(\alpha^2 + 1) + (1 - \alpha^2) \cos 2\lambda P t} \quad (9-4)$$

Where α and λ are defined in equation (8-2), and the plus and minus signs refer to "Max I" and "Min I" vehicles, respectively. For symmetric vehicles, where $\alpha = 1$, the rotation rate of the wobble vector and in turn ω_G are both equal to the wobble frequency, λP . The total torque impulse, I_T , applied in opposition to the wobble, per inertial wobble period, is given by,

$$I_T \equiv \int_0^{2\pi} |T_{C_T}| d\tau = (P + \omega_G) h \left(\frac{1}{P \pm \lambda P} \right) \quad (9-4)$$

For a symmetric vehicle $\omega_G = \pm \lambda P$, so that

$$I_T = h, \quad (9-5)$$

which is the maximum deliverable angular momentum.

A previous analysis performed at Grumman (Reference 10) proved, using an energy sink method, that the 90 degree-H-Lag Law is time optimum for a symmetric vehicle and is an averaging approach to the time optimal law for non-symmetric vehicles. In fact, for most vehicles, symmetric or unsymmetric, the damping time penalty associated with 90 degree-H-Lag Law is negligible. Furthermore, to mechanize the true optimal law would not be practical (as noted in Reference 10).

In order to stop the application of control torques, the CMG gimbal is driven so that the CMG momentum vector is stationary in inertial space. That is, when the CMG is located in the rotating portion of the vehicle it is driven at a rate equal to minus the spin rate. The stop criterion is based upon whether the wobble amplitude is within the Target Ellipse, or not. The semi-major and semi-minor axes of the Target Ellipse are defined as $\alpha \Omega_r$ and Ω_r , respectively; for $\alpha < 1$, the definitions are reversed.

Furthermore,

$$\Omega_r = \begin{cases} \frac{\omega_r}{\alpha}, & \text{for } \alpha \geq 1 \\ \omega_r, & \text{for } \alpha < 1, \end{cases} \quad (9-6)$$

Where the values for the maximum allowable wobble levels (ω_r), corresponding to both "T" and "Y" configurations, are given in Table 5.2.

The basic CMG wobble damping concept, as presented in the above paragraphs evolved from rigid body analyses. Applying this basic concept to the flexible space station resulted in interactions with the structure which necessitated a few modifications. The results of the flexible body analyses are presented in Section 9.2.

Before proceeding to the flexible body analysis, an estimate for the required CMG wheel size (h) is obtained. The wobble damping task is specified by stipulating that the amplitude of a component of wobble rate (Ω) is to be reduced within a specified time (t_f) to a value below a specified threshold, $\Omega(t_f)$. Based upon a rigid body analysis, Reference 10 shows that the 90 degree-H-Lag-Law control

concept will require a wheel size that is given by,

$$h = \frac{\Omega(0) - \Omega(t_f)}{K t_f} \quad (9-7)$$

where

$$K = \frac{2I_x \lambda P \bar{G}(\alpha)}{\pi I_z |I_x - I_z|}$$

$$\bar{G}(\alpha) \equiv \begin{cases} F(k); & 1 \leq \alpha \leq \infty, k = \frac{1}{\alpha} (\alpha^2 - 1)^{\frac{1}{2}} \\ \alpha F(k^1); & 0 \leq \alpha \leq 1, k^1 = (1 - \alpha^2)^{\frac{1}{2}} \end{cases}$$

$F(k) \equiv$ the complete elliptic integral of the first kind.

All other variables are defined as in Section 8.1.

Applying the above relationships to the nominal "T" and "Y" configurations (as defined in Section 4.0) in combination with the wobble damping requirements as specified in Section 5.1 results in the following specifications for the CMG wheel size.

Configuration	Required Wheel Size h (ft-lb-sec)
"Y" - in response to internal mass motion ¹	385
"T" - in response to internal mass motion ¹	9,670
"Y"/Shuttle - in response to one sigma docking ²	11,170
"T"/Shuttle - in response to one sigma docking ²	17,870

1 Mass motion is defined in Reference 7. Initial disturbance is presented in Section 5.2.

2 Docking disturbance is derived in Section 5.2.

9.2 Wobble Damping of the Flexible Space Station

The CMG wobble damping concept, previously described, was applied to the General Flexible Body Simulation - Stage II. A functional block diagram of the math model for the torquer motor and gimbal dynamics along with the associated control logic is shown in Figure 9-2. The natural frequency and damping for the torquer-gimbal combination used for all studies were 3 rad/sec and 0.8, respectively. The components of the applied control torque are derived in the simulation as follows:

$$\begin{aligned}
 T_{c_1} &= h(\omega_3 \sin \beta + \omega_2 \cos \beta) \\
 T_{c_2} &= -h(\omega_1 + \dot{\beta}) \cos \beta \\
 T_{c_3} &= -h(\omega_1 + \dot{\beta}) \sin \beta
 \end{aligned}
 \tag{9-8}$$

where

$\omega_1, \omega_2, \omega_3$ - are vehicle angular rates of the body that the CMG is within
 $\beta, \dot{\beta}$ - CMG gimbal angle, rate. Where β is measured positively from axis 2, and $0 \leq \beta \leq 2\pi$.

Nominal values for all the physical vehicle properties (refer to Section 4.0) were used during the flexible body analyses with CMG control. The initial disturbance, for CMG sizing purposes, is the result of a particular mass motion profile corresponding to a 2,000 lb mass moving within rotor arm #1. The initial transverse angular rate resulting from this disturbance was derived in Reference 7 and presented in Section 5.2 for completeness.

Applying the basic CMG wobble damping concept to the "Y" space station resulted in successful wobble damping without modification to the control system. A low pass filter, with a two second time constant, was added to the system so as to avoid unnecessary interaction with the structure and in turn decrease the actuator duty cycle. Figure 9-3 are the resulting time history traces with a 400 ft-lb-sec CMG placed in the "Y" rotor. It is noted that all motion

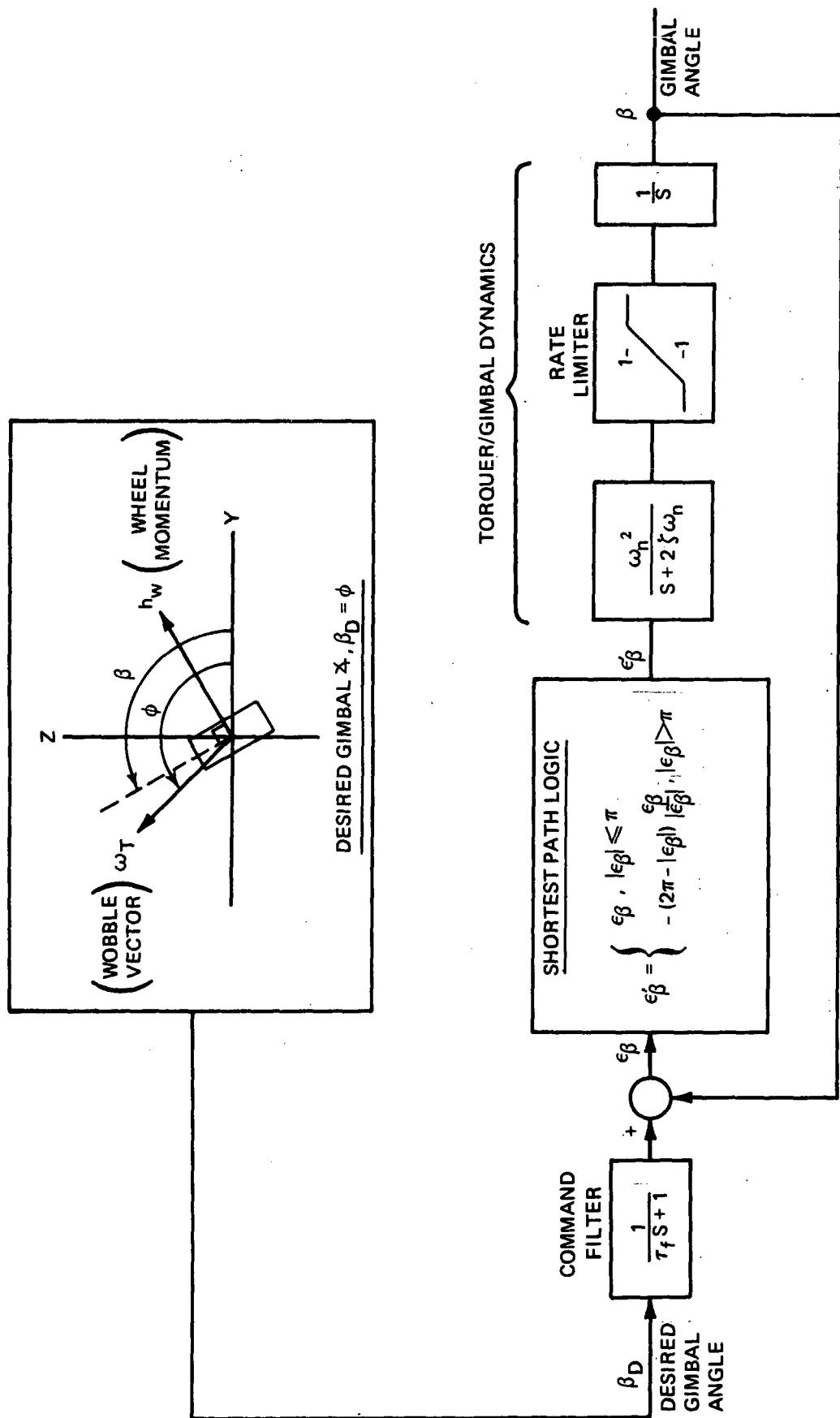
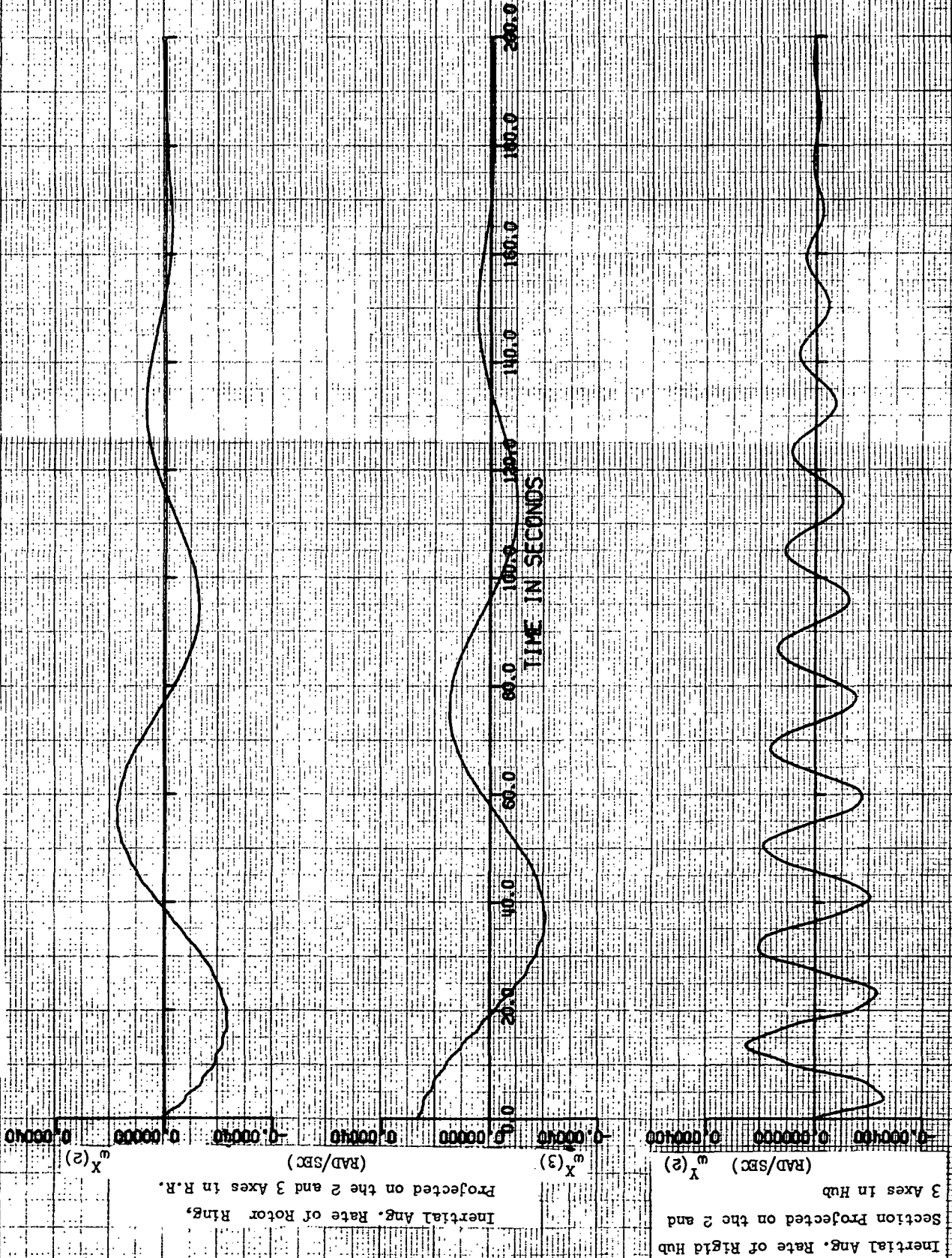


Figure 9-2. CMG Wobble Damping Logic And Simulated Dynamics

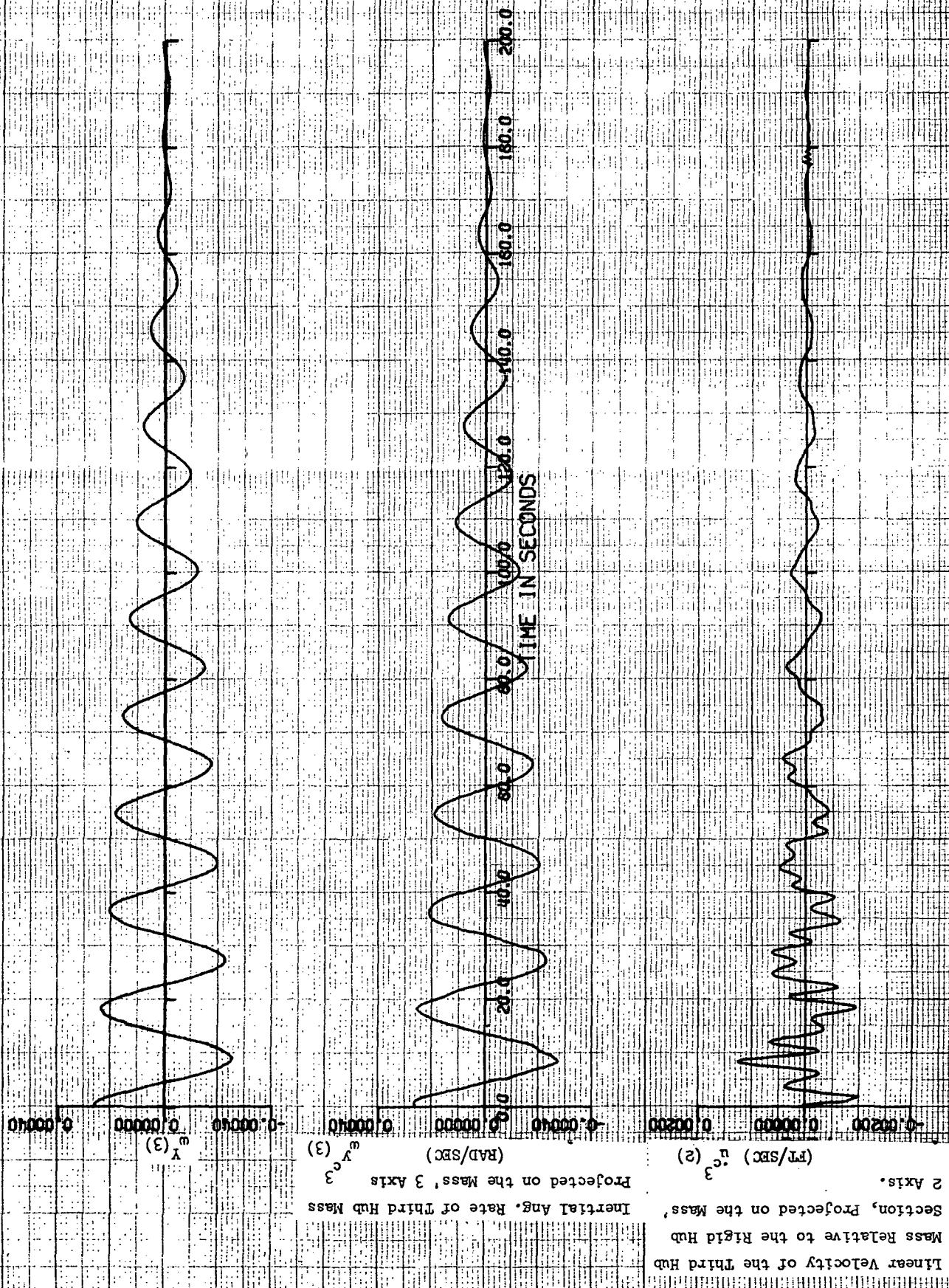
FIGURE 9.3 "Y" - NORMAL, CMG CONTROL (H = 400 FT-LB-SEC)



16-633-23

27-833-23

FIGURE 9.3 (Cont)



11-60312D

FIGURE 9.3 (Con't.)

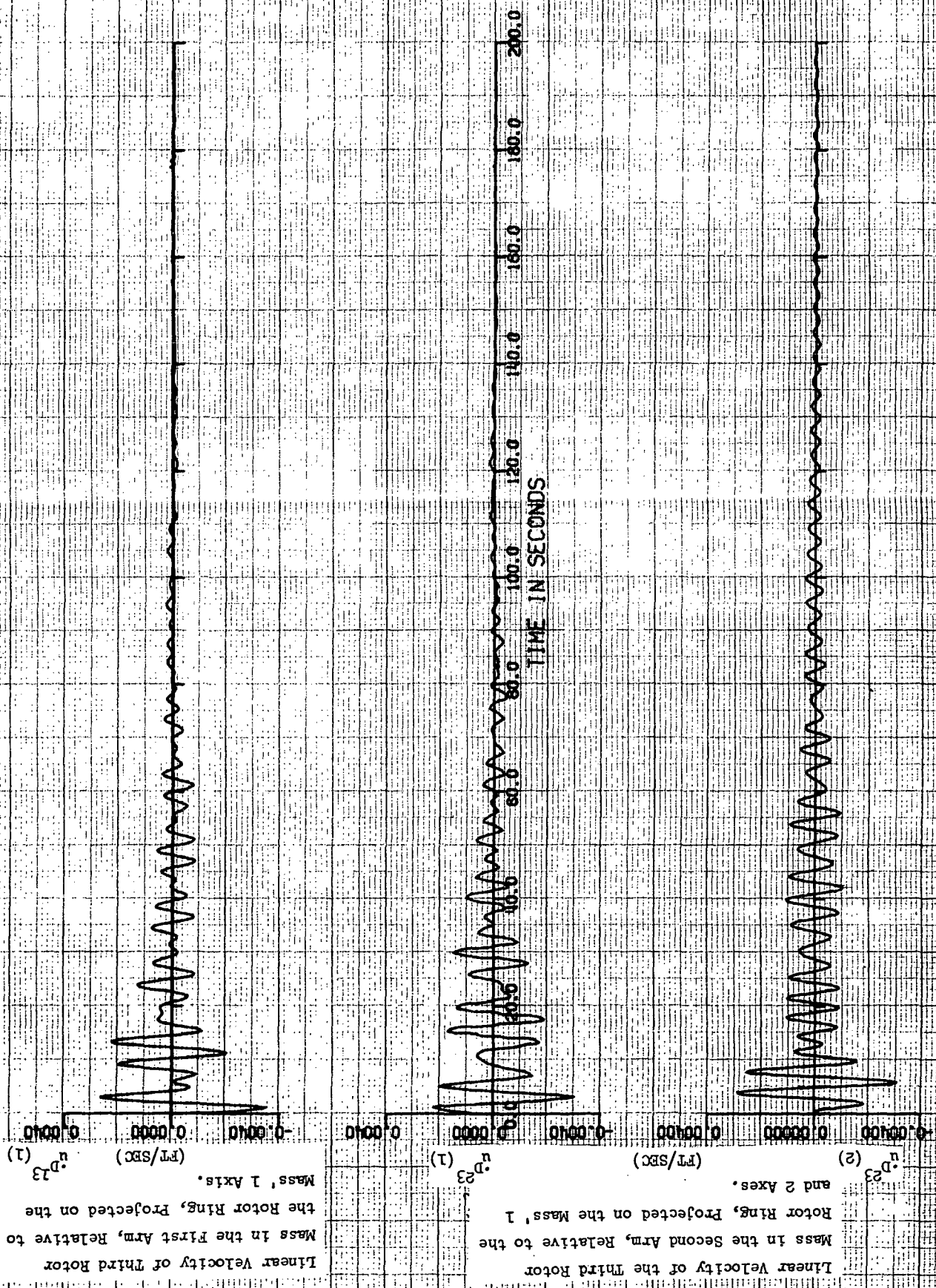
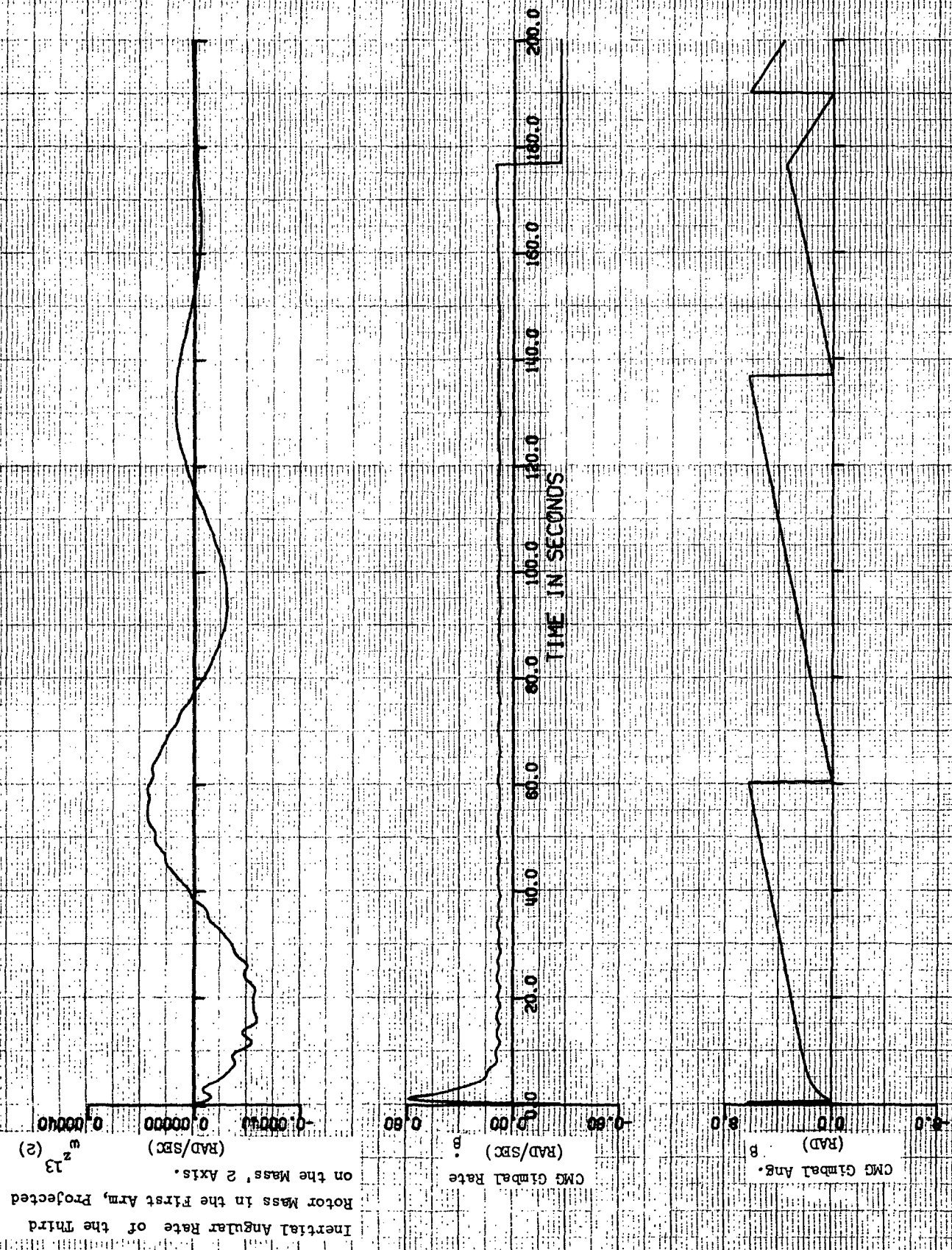


FIGURE 9.3 (Cont'd)



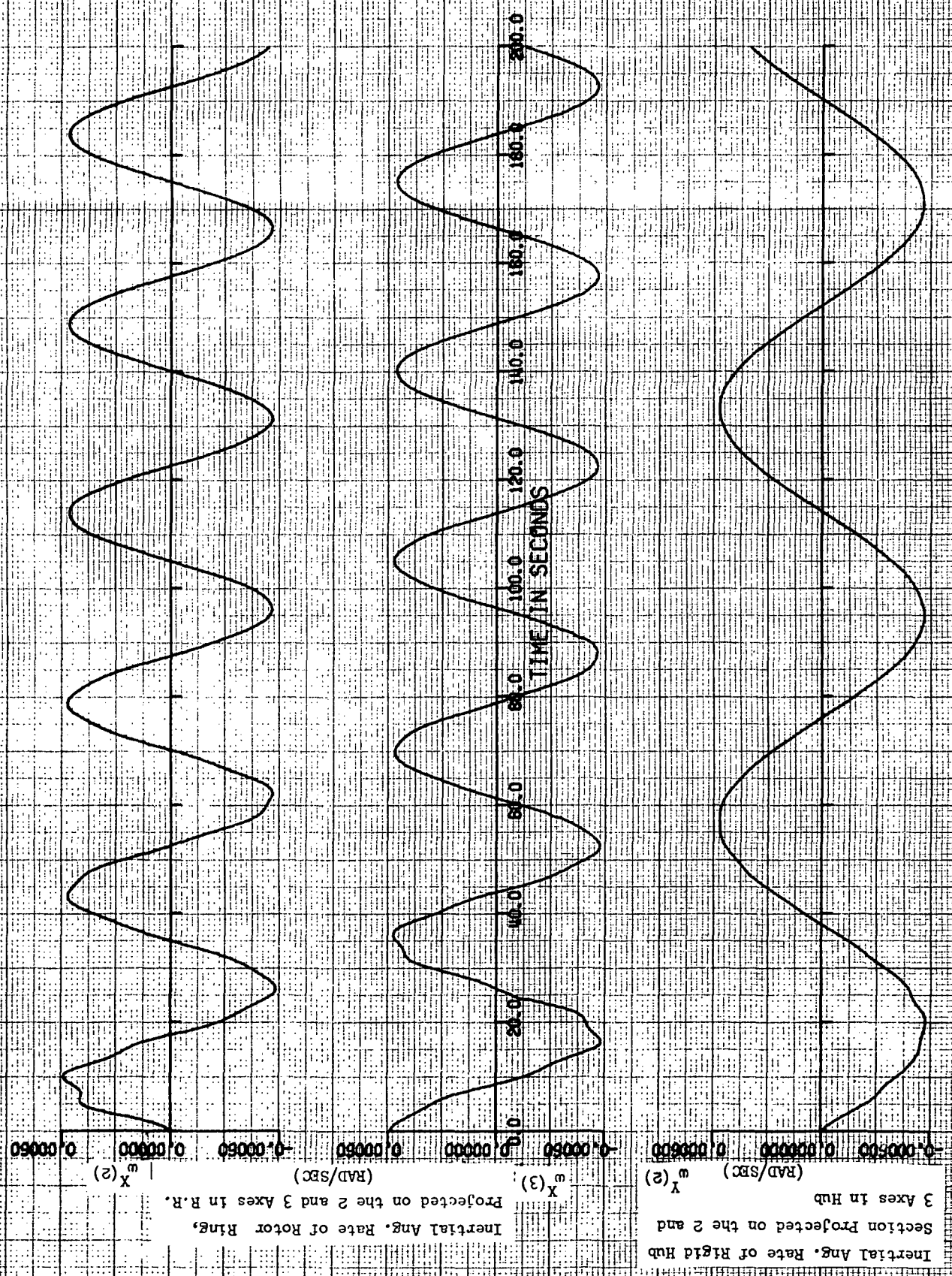
is damped including the relative flexible motion as exemplified by the traces of relative linear velocities (\dot{u}^D and \dot{u}^C). Referring to the time history for gimbal rate ($\dot{\beta}$) it is seen that the wobble damping task is satisfied within the required time (180 sec). For example, when the desired wobble level is reached, the CMG is driven at minus spin rate. Although the gimbal is 90 degrees out of phase at the start of the simulation, the CMG gimbal is at its desired position within 10 seconds and maintains track until the wobble is damped within the desired level.

Applying the same CMG system to a "Y" station with the Shuttle attached, and disturbed by a 1 sigma docking, results in the time histories shown in Figure 9-4. Although the gimbal does lock-on and track the wobble vector, the amount of damping during the run duration is slight due to the small CMG wheel size (400 ft-lb-sec). It is estimated that it would take 5,000 seconds to completely damp the wobble induced by the docking using a 400 ft-lb-sec CMG.

It should be noted that it is considered unreasonable to size the CMG to accommodate the relatively infrequent docking disturbances. If faster damping is required in response to a docking, then the reaction jet system can be used.

Unlike the immediate successful results experienced with the "Y" station, the application of the basic wobble damping concept to the "T" station rotor required modifications for proper operation. The first difficulty encountered was a high frequency limit cycle resulting from the gimbal rate being less than minus spin rate. It is noted that a rigid "T" station requires a gimbal rate close to negative spin and with the addition of small flexible motions there are periods of a commanded gimbal rate below negative spin. The result of this phenomenon is a reversal of the direction of rotation of the wheel angular momentum vector in inertial space; in turn, control torques are aiding the wobble rather than opposing it. It should be noted that for any rigid body vehicle this anomaly would not occur. In order to fix this problem a negative rate limit equal to minus the spin rate is incorporated.

FIGURE 9.4 "Y" / SHUTTLE - NOMINAL, CMG CONTROL (H = 400 FT-LB-SEC)



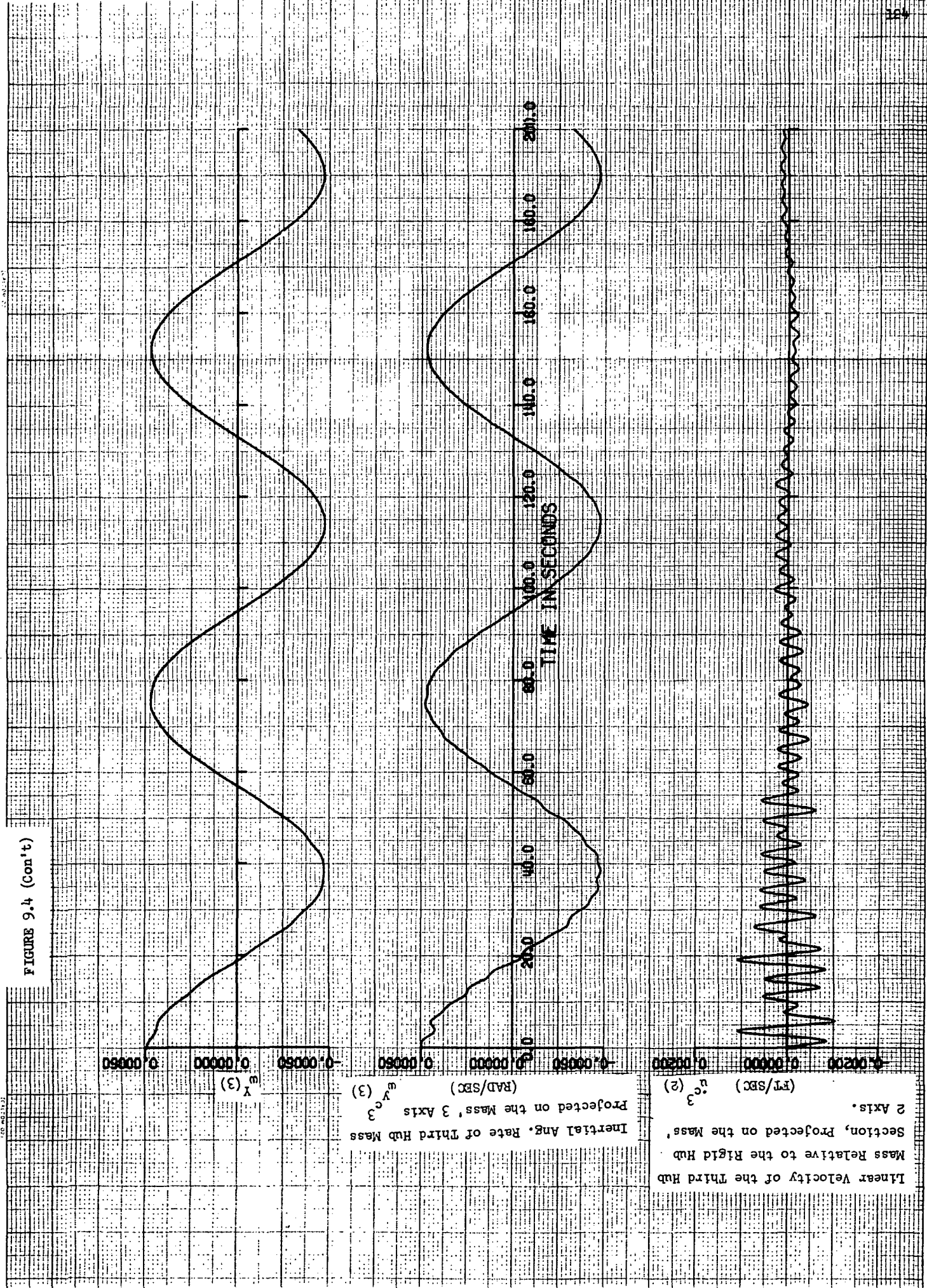


FIGURE 9.4 (Cont.)

15-58112-2

FIGURE 9.4 (Con't)

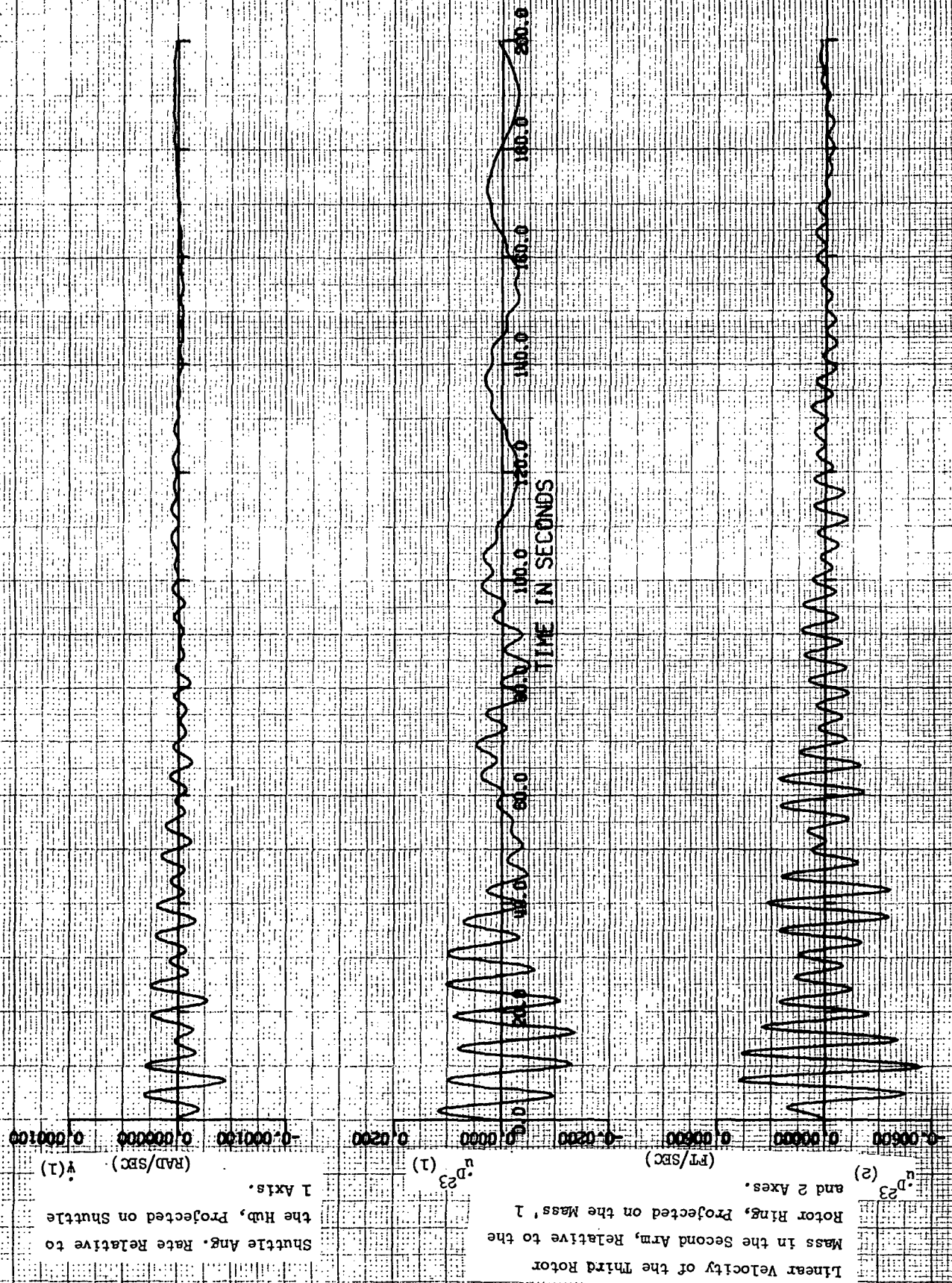


FIGURE 9.4 (Con't)

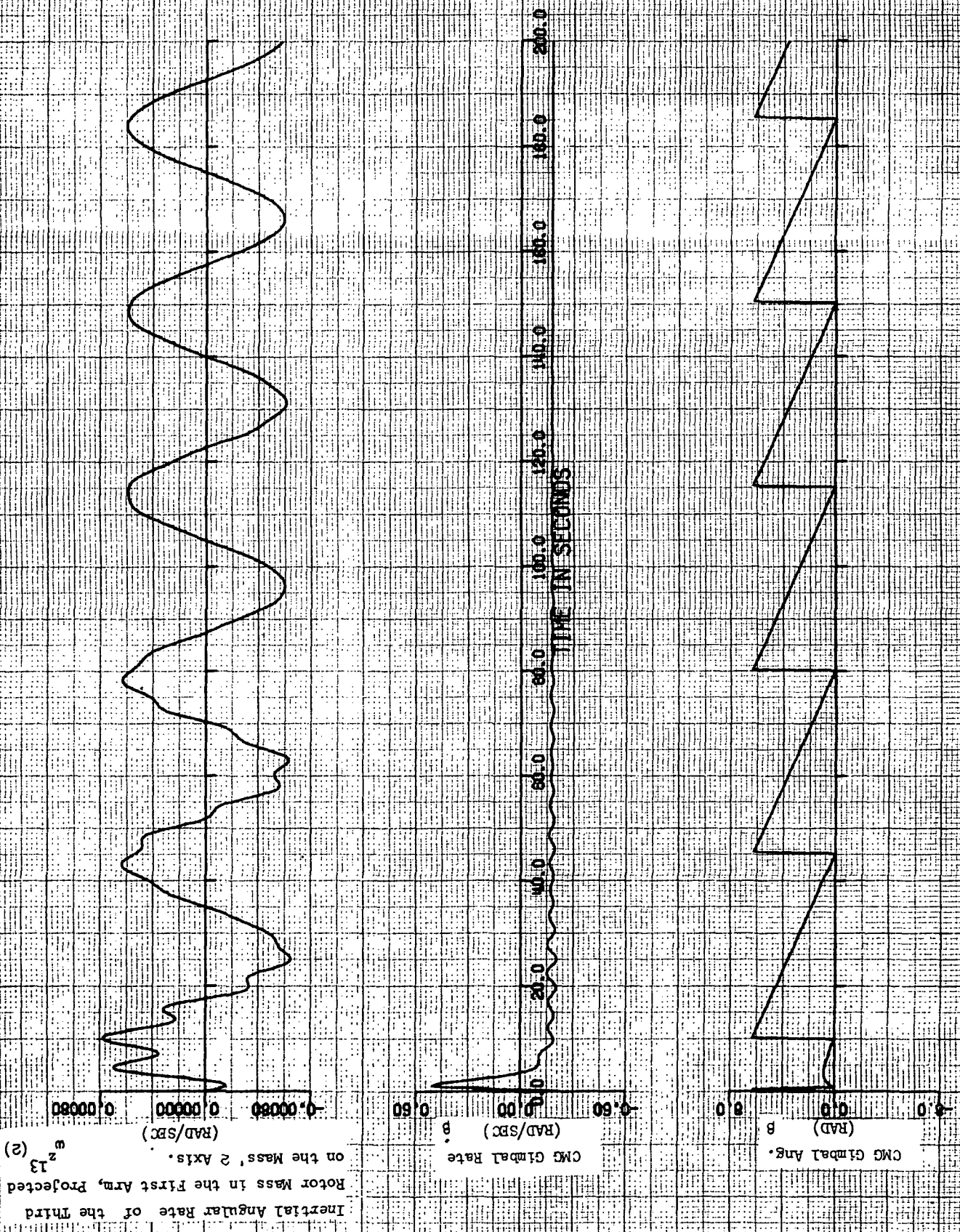


Figure 9-5 contains "T" configuration time history traces resulting from the application of the basic CMG system with the additional negative rate limit, a 10,000 ft-lb-sec CMG, and a low pass filter with a 2 second time constant. Two undersirable effects are noted:

1. The wobble damping task is not accomplished within the required 180 seconds.
2. There is a significant amount of control interaction with the structure as indicated by the activity on the $\dot{\beta}$ trace.

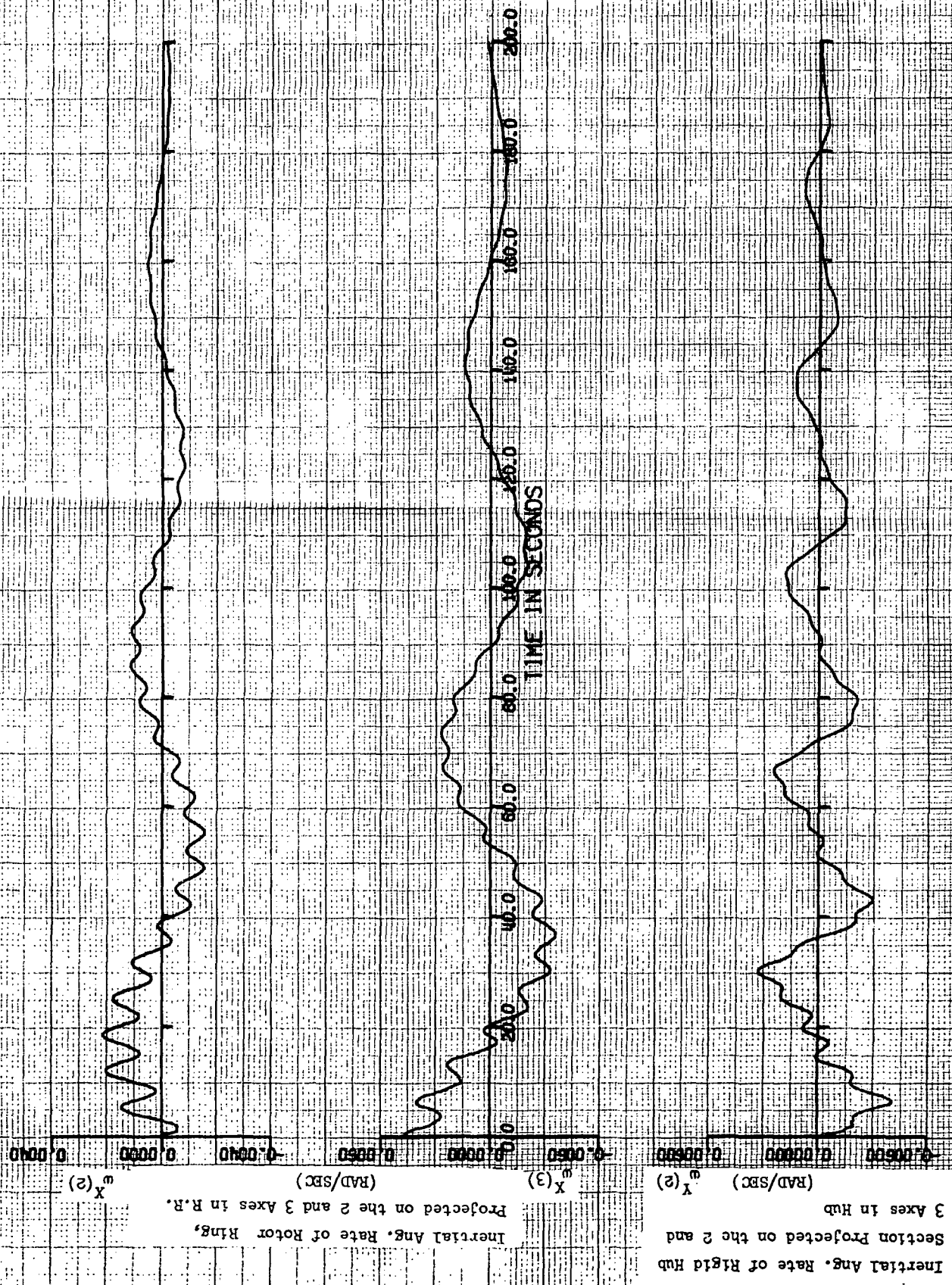
The solution to the first problem is to increase the wheel size. A 12,000 ft-lb-sec CMG was shown to be sufficient. Although there was no structural instability observed as a result of item 2, the possibility for an instability does exist as shown in Reference 3. In addition, high frequency oscillation of the CMG should be avoided from a wear point of view. In order to avoid control/structural interaction, the filter bandwidth, decreased. However, as the bandwidth of the filter is decreased, the tracking error is increased, which results in an inefficiency. As a result, the recommended final modifications for the "T" configuration CMG system are as follows:

- o increase the filter time constant to 10 seconds.
- o add a gimbal angle command bias to compensate for the additional tracking error.

A good approximation for the tracking error and in turn the gimbal command bias is, $(\Delta P)_{T_f}$, the wobble frequency times the filter time constant. The time history traces resulting from the application of the CMG system with all of the above mentioned modifications are shown in Figure 9-6. As indicated by these time histories, the wobble damping task is successfully accomplished. It should be noted that the criterion for stopping active CMG damping was rased slightly in order to avoid interaction with the structural modes at low wobble amplitudes.

Figure 9-7 contains time history traces resulting from the application of the final CMG system, with all required modifications, to a "T" station with an attached shuttle. The initial disturbance is equivalent to a 1 sigma docking. Although wobble damping is

FIGURE 9.5 "n" - NOMINAL, CMG CONTROL (H = 10,000 FT-LB-SEC)



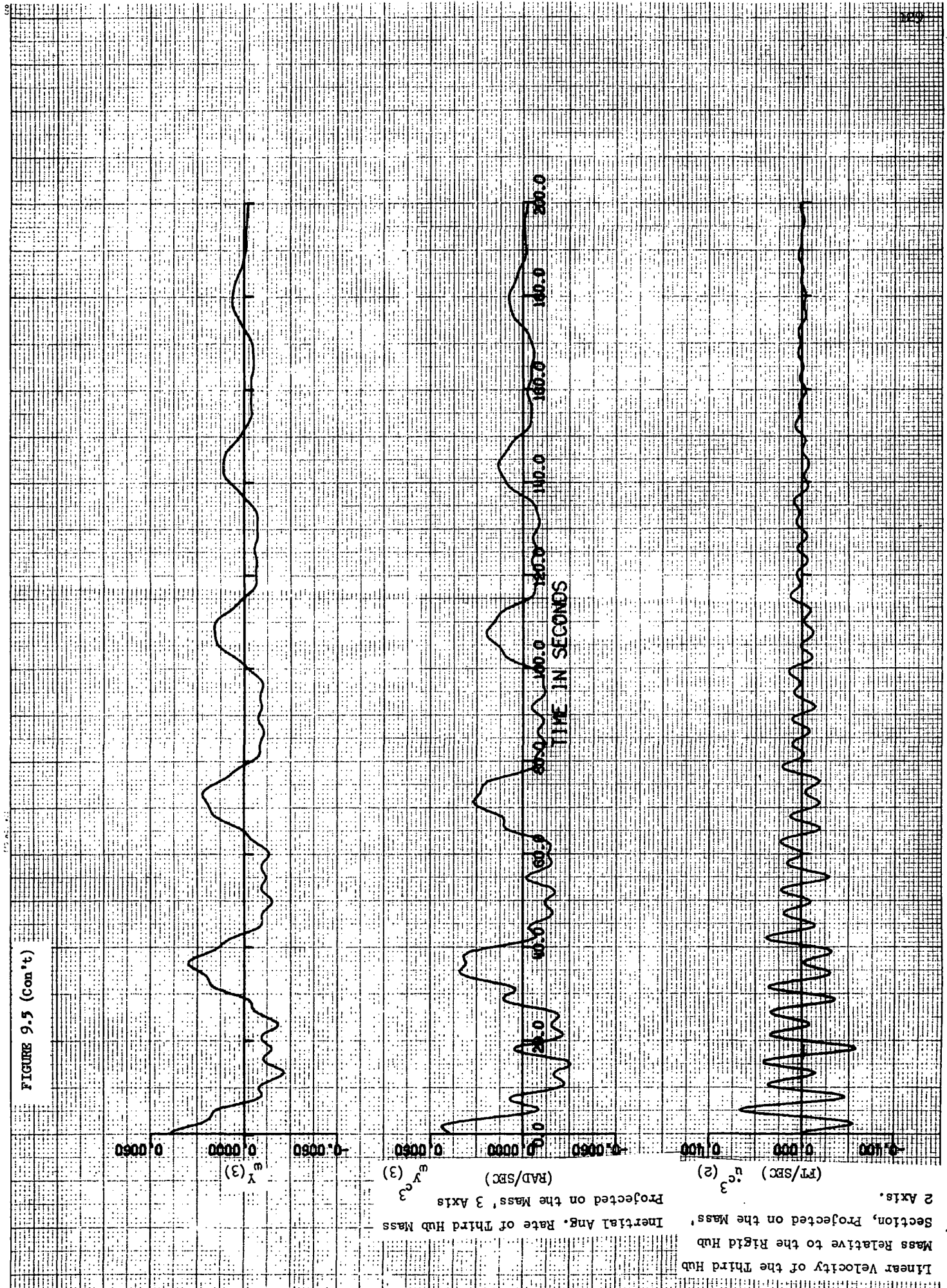


FIGURE 9.5 (Con't)

Linear Velocity of the Third Hub Mass Relative to the Rigid Hub Section, Projected on the Mass' 2 Axis.

Inertial Ang. Rate of Third Hub Mass Projected on the Mass' 3 Axis

m^3 (3)

(FT/SEC) m^3 (2)

(RAD/SEC) m^3 (3)

0.0050 0.0000 -0.0050

0.1000 0.0000 -0.0050

0.0050 0.0000 -0.0050

0.0050 0.0000 -0.0050

TIME IN SECONDS

200.0

140.0

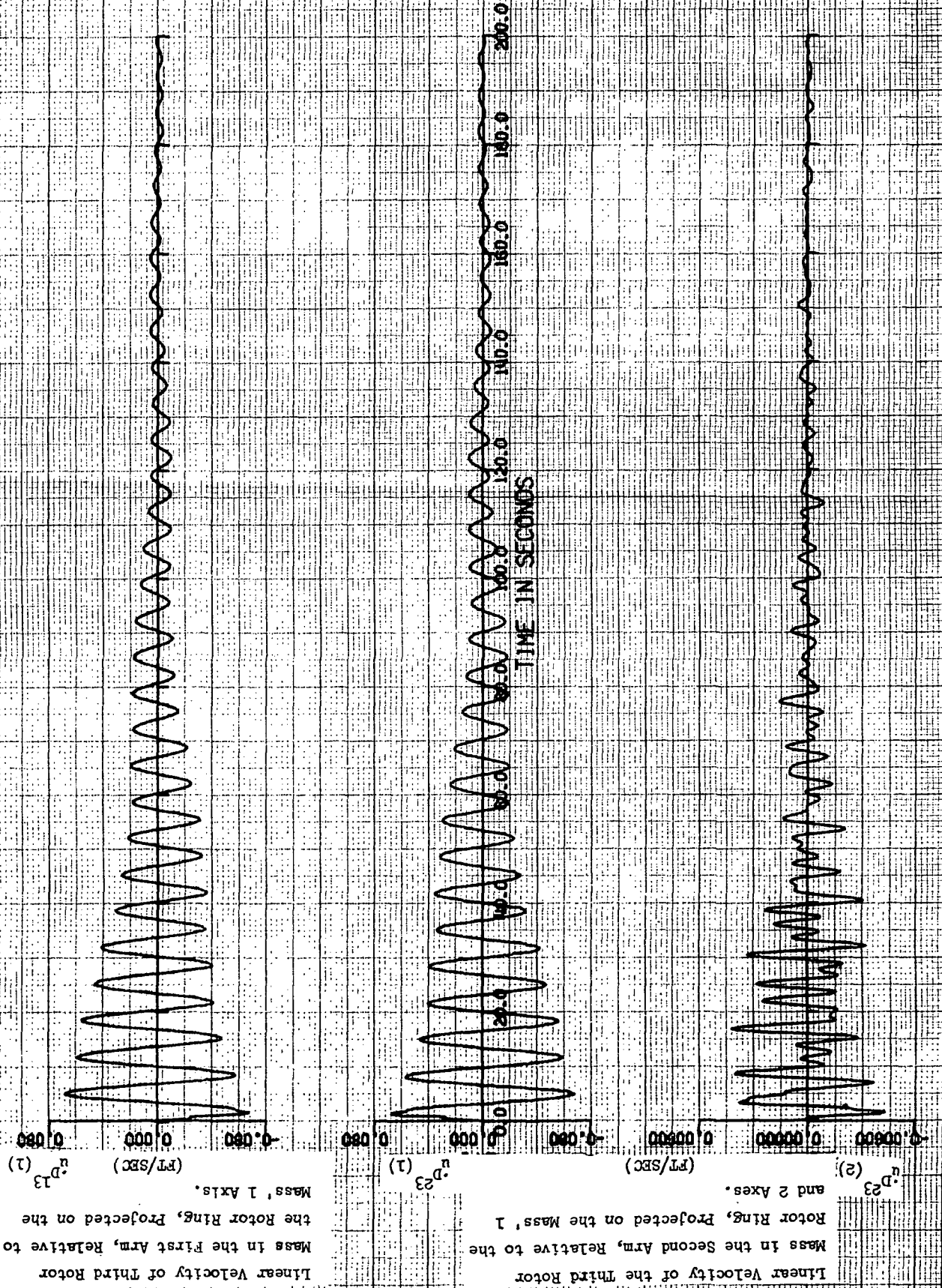
100.0

60.0

20.0

0.0

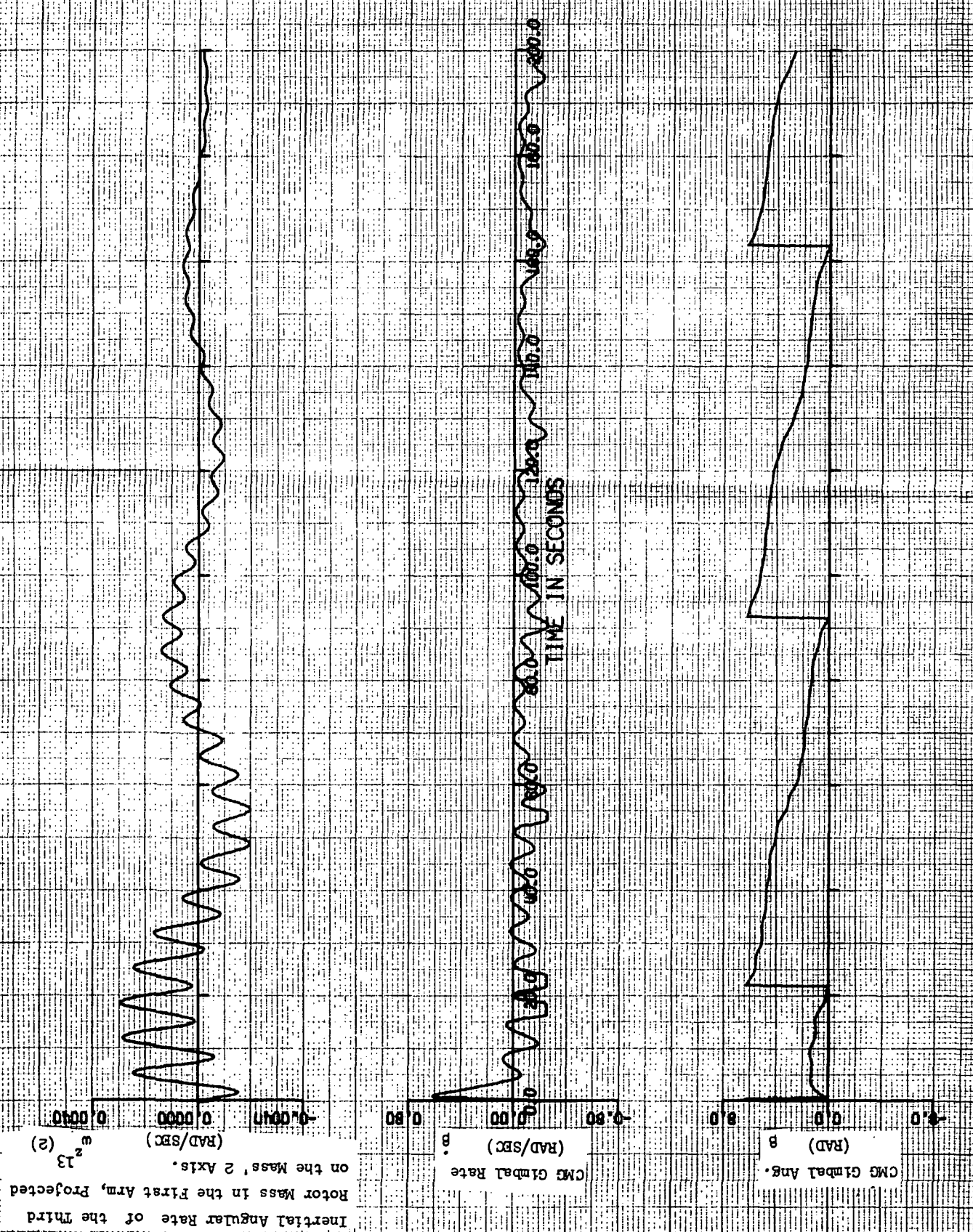
FIGURE 9-5 (Con't)



Linear Velocity of Third Rotor
Mass in the First Arm, Relative to
the Rotor Ring, Projected on the
Mass' 1 Axis.

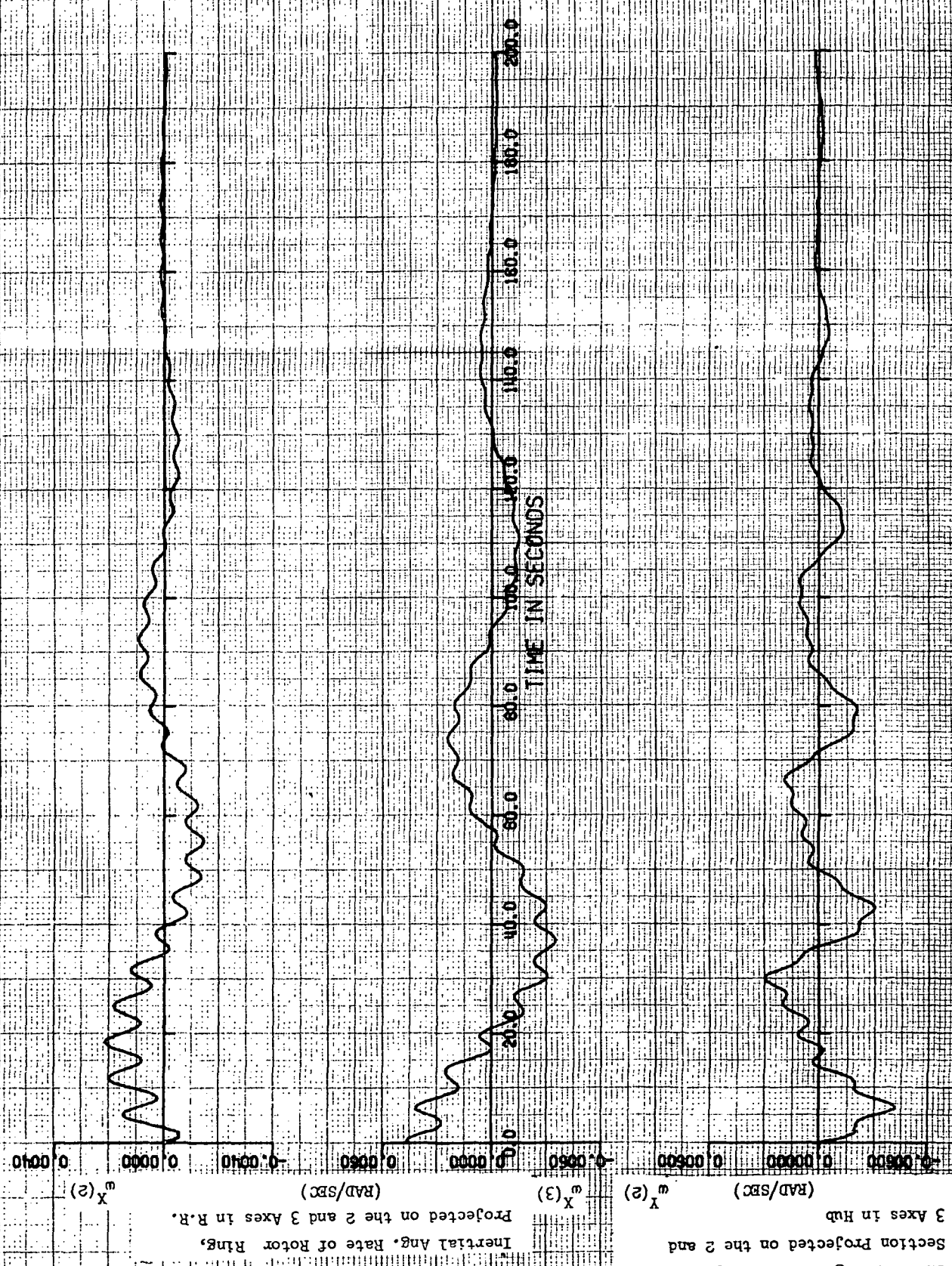
Linear Velocity of the Third Rotor
Rotor Ring, Projected on the Mass' 1
and 2 Axes.

FIGURE 9-5 (con't)



AS 11-1

FIGURE 9.6 "T" - NOMINAL, CMG CONTROL
 (H = 12,000 FT-LB-SEC, $\tau_f = 10$ SEC,
 GIMB ANG BIAS)



Inertial Ang. Rate of Rigid Hub Section Projected on the 2 and 3 Axes in Hub

$\omega_X(2)$

$\omega_X(3)$

(RAD/SEC)

$\omega_X(2)$

Inertial Ang. Rate of Rotor Ring, Projected on the 2 and 3 Axes in R.R.

$\omega_X(2)$

(RAD/SEC)

TIME IN SECONDS

200.0

180.0

160.0

140.0

120.0

100.0

80.0

60.0

40.0

20.0

0.0

0.00500

0.00000

-0.00500

(RAD/SEC)

$\omega_X(2)$

-0.00500

0.00000

0.00500

(RAD/SEC)

$\omega_X(2)$

-0.00500

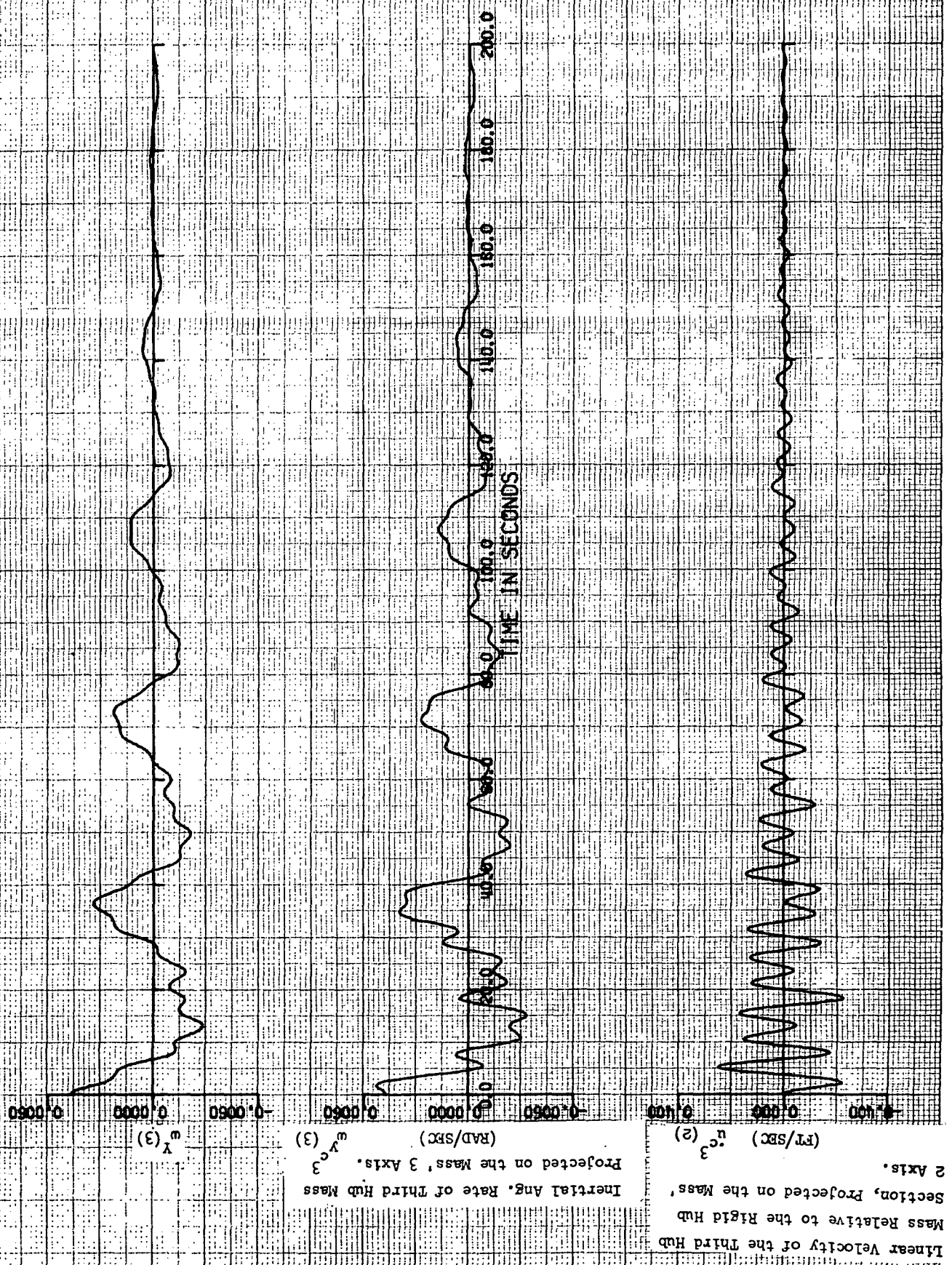
0.00000

0.00500

(RAD/SEC)

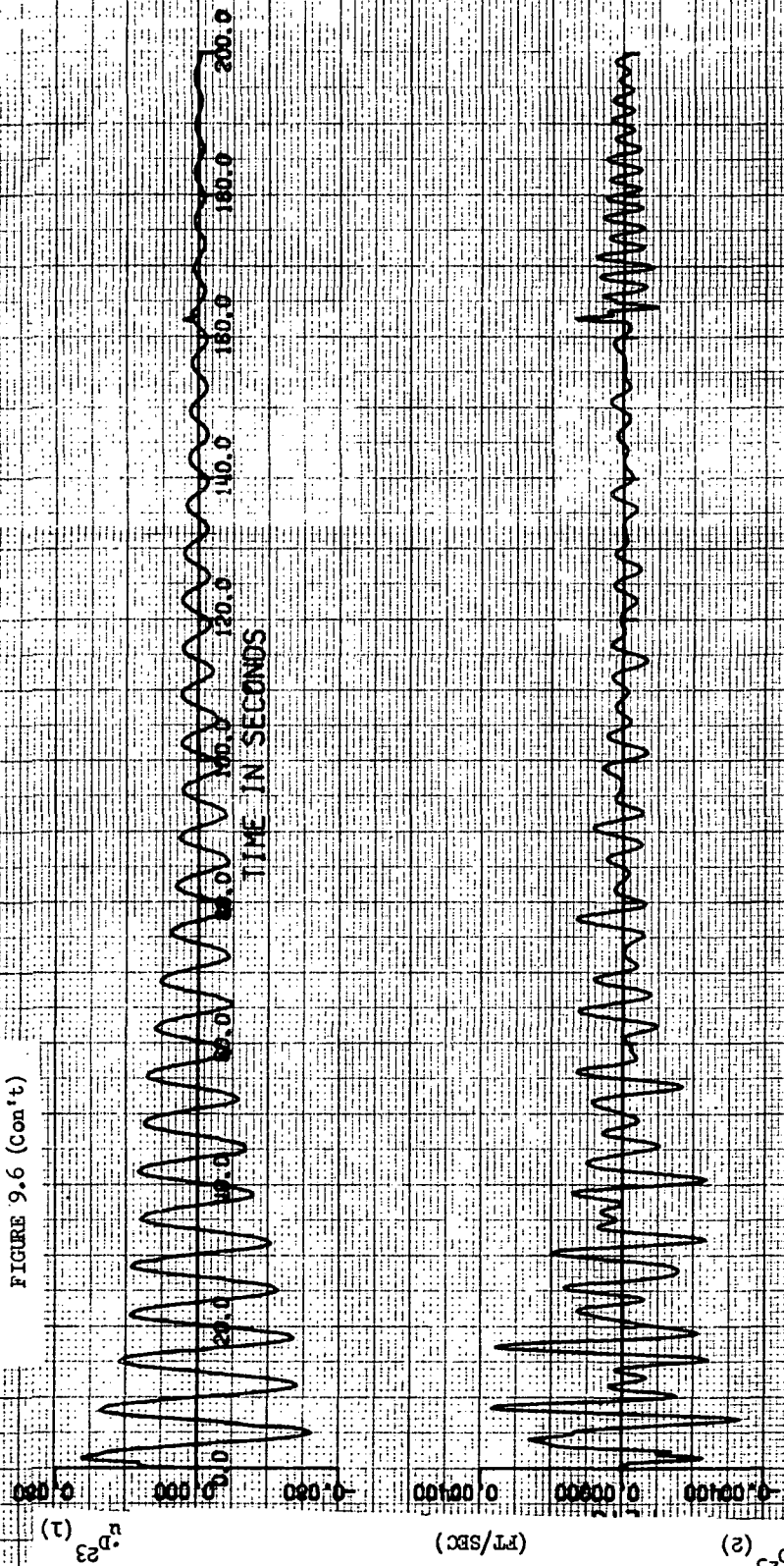
$\omega_X(2)$

FIGURE 9.6 (Con't)



CG 25-7531

FIGURE 9.6 (Cont.)



Linear Velocity of the Third Rotor
Mass in the Second Arm, Relative to the
Rotor Ring, Projected on the Mass' 1
and 2 Axes.

FIGURE 9.6 (Con't)

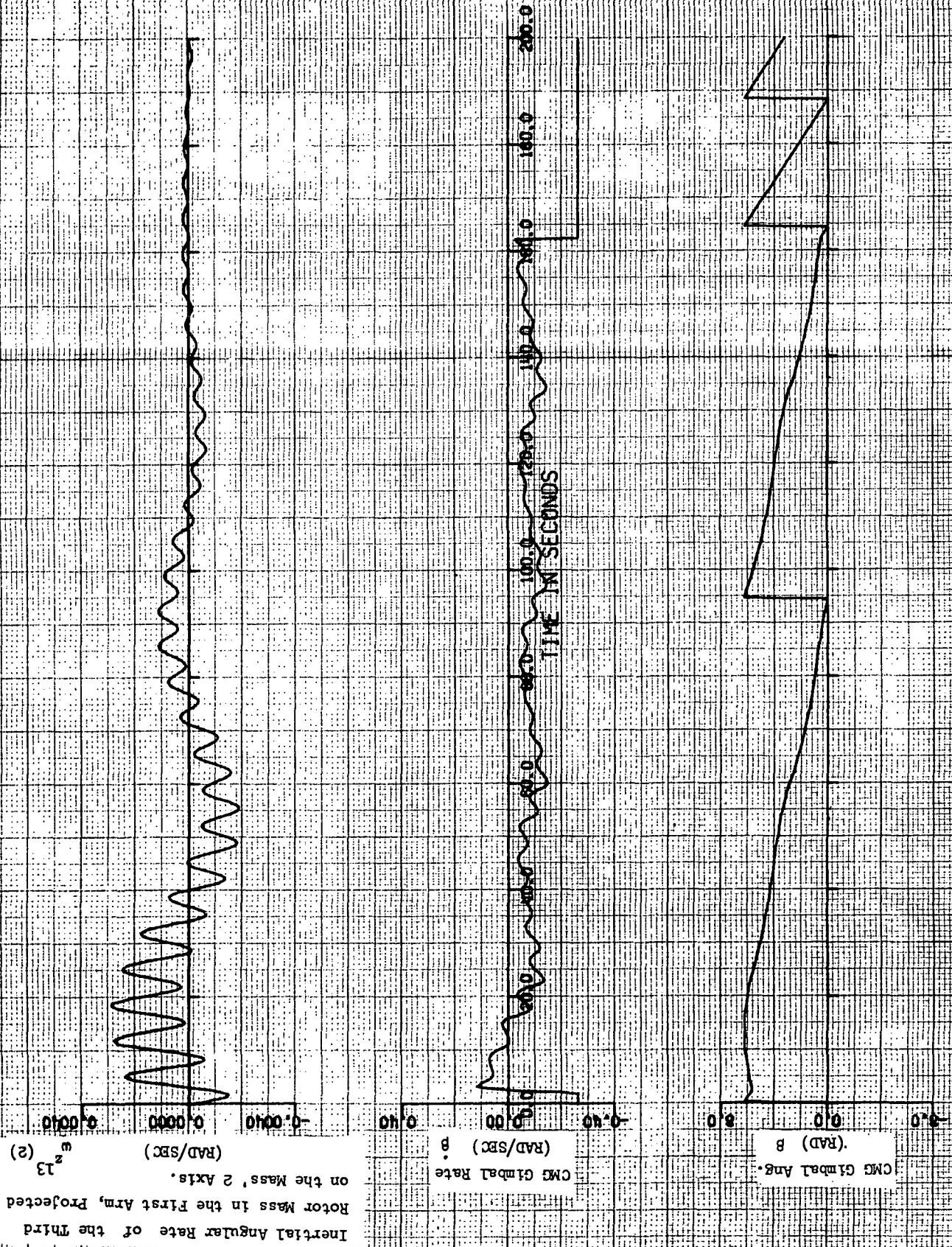


FIGURE 9.7
"T" SHUTTLE - NOMINAL, CMG CONTROL
(H = 12,000 FT-LB-SEC, $\tau_f = 10$ SEC,
GIMB ANG BIAS)

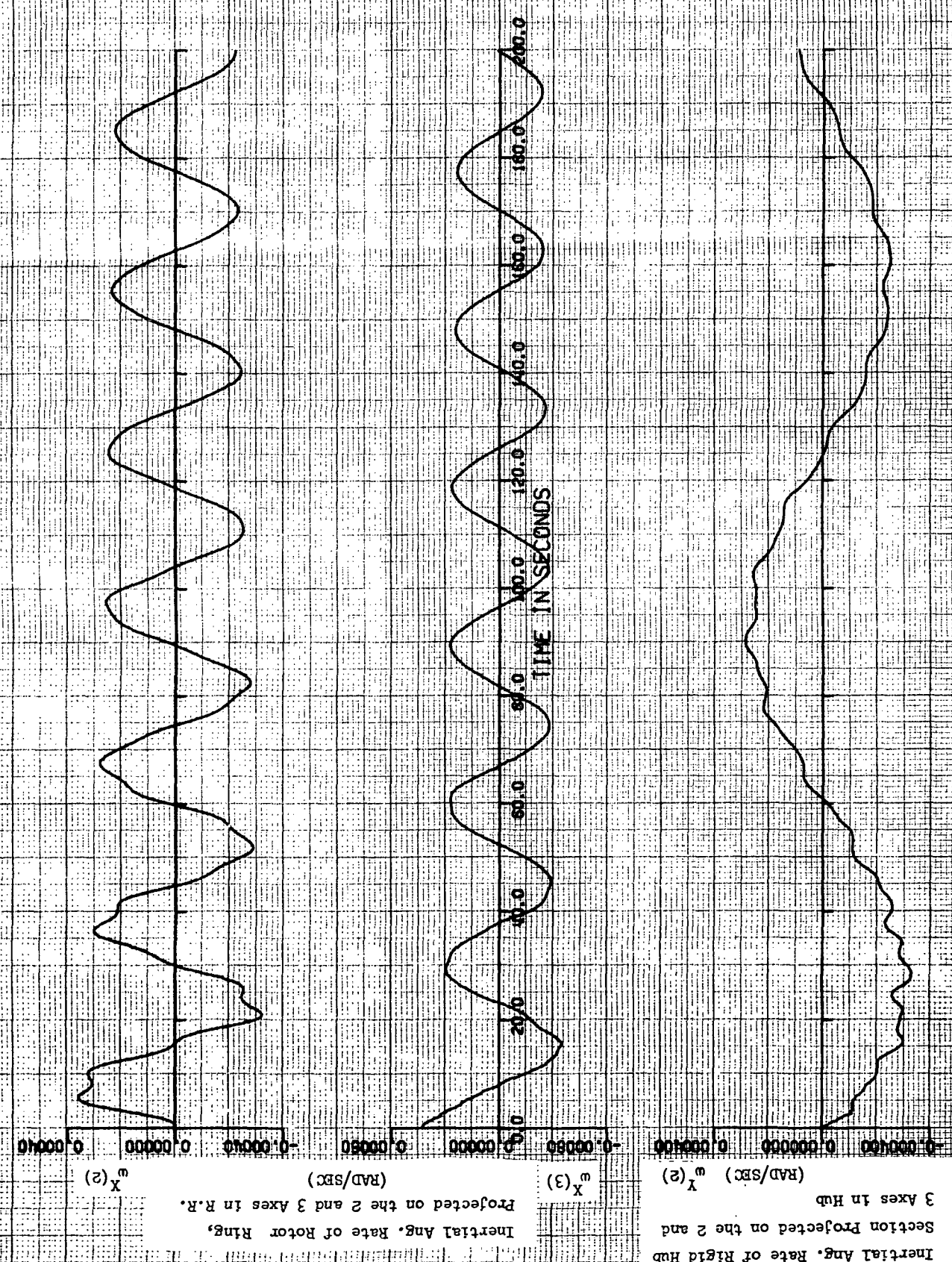


FIGURE 9.7 (con't)

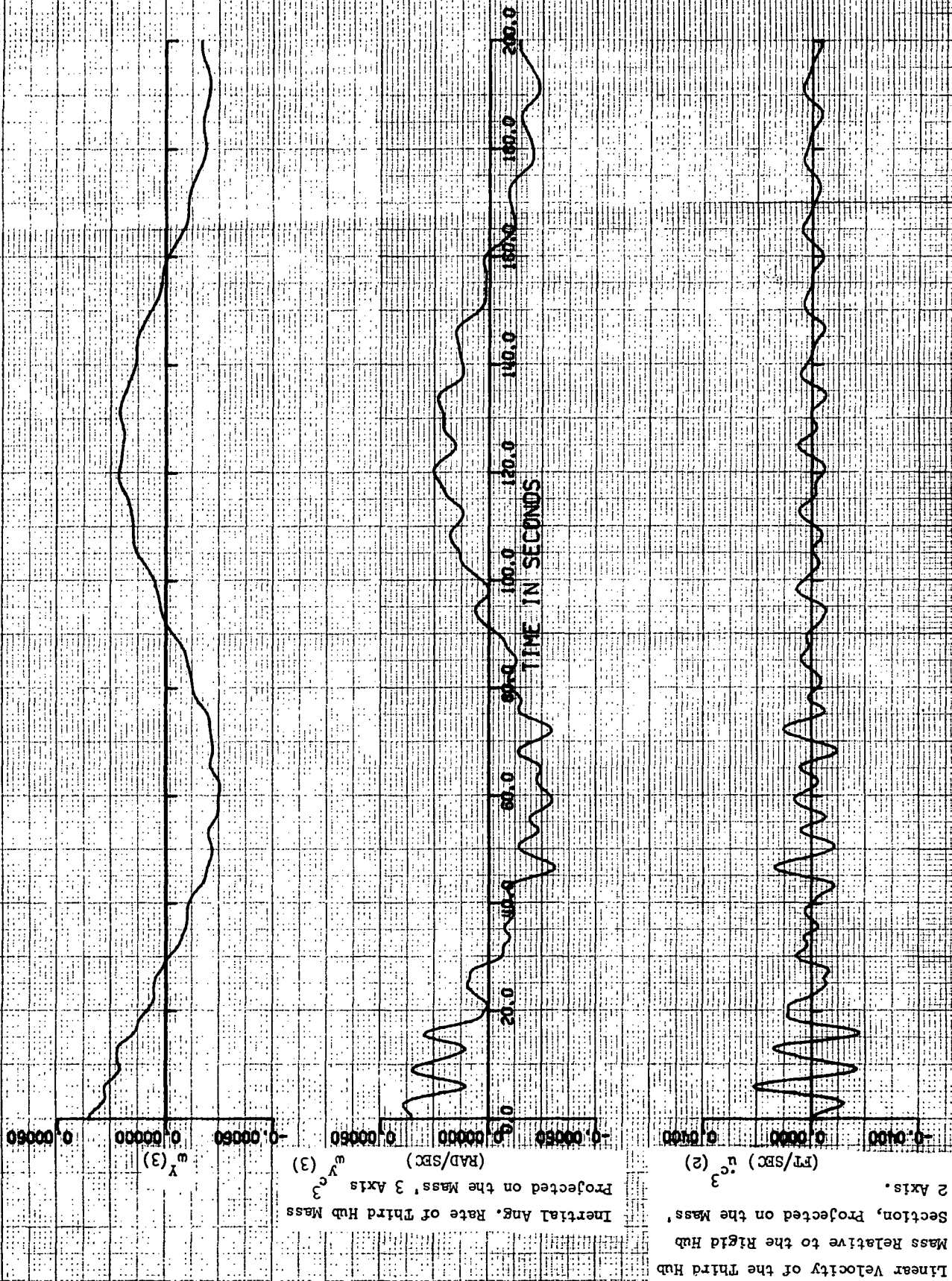


FIGURE 9.7 (Con't)

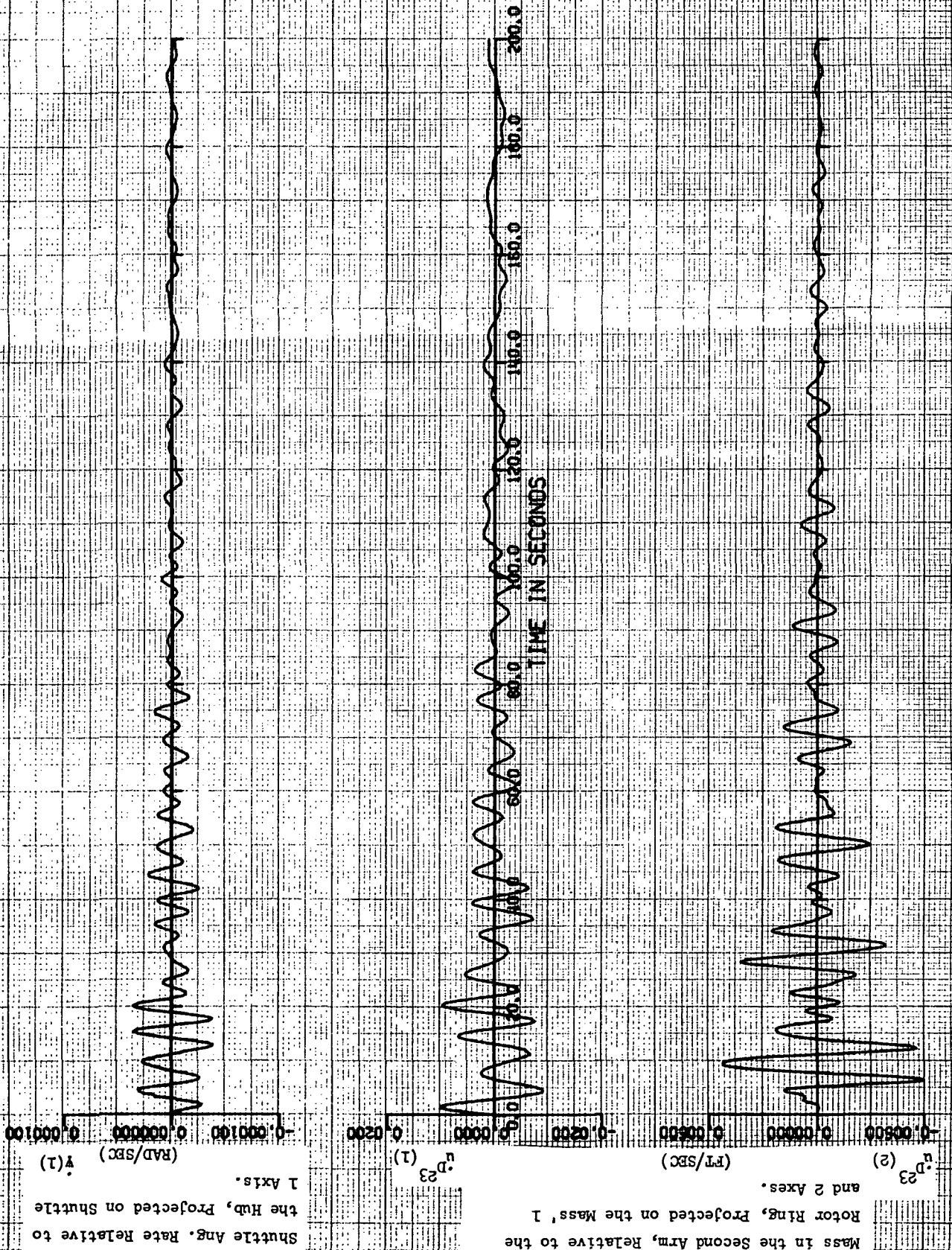
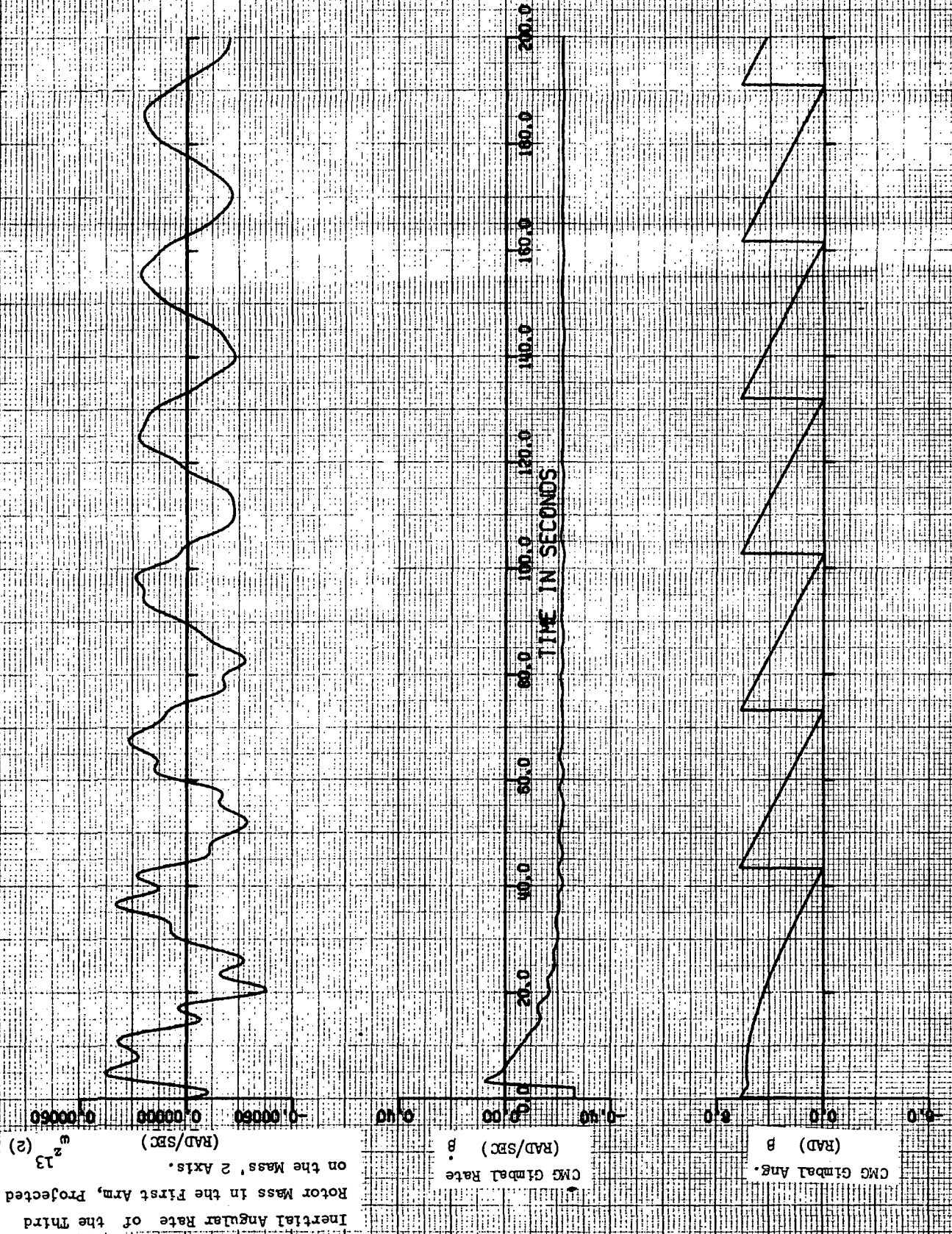


FIGURE 9.7 (Con't)



observable, it noted that the wheel size is not sufficient to complete the damping task within the specified time. However, as previously stated, CMG sizing will not be based upon infrequent docking disturbances.

9.3

CMG Requirements

Summarizing the results of the flexible body simulation studies, the following table lists the CMG system requirements.

TABLE 9-1
CMG System Requirements

Configur- ation	Wheel Size (ft-lb-sec)	Filter, Nat. Freq, (rad/sec)	Command Bias (rad)	<u>Torquer/Gimbal</u>		
				Nat. Freq. (rad/sec)	Damp Const.	Max. Average Output Power (watts)
"Y"	400	0.5	--	3	0.8	.009
"T"	12,000	0.1	-0.97	3	0.8	5.15

The output power, P_{out} , is derived by using,

$$P_{out} = T_{c1} \omega_G \quad (9-9)$$

Applying $T_{c1} = |\omega_T| h$, along with equation (9-4) for the gimbal rate, and averaging over the period of the cyclic portion of ω_G , results in

$$(P_{out})_{AV} = |\omega_T| h (\lambda P) \frac{2\alpha}{(\alpha^2 + 1)} \quad (9-10)$$

As noted from equation (9-10), the average output power will vary with the wobble rate magnitude $|\omega_T|$. The maximum average power is defined as the average output power required at the beginning of the wobble damping task. Using actual values for α and wobble frequency (λP), as experienced during the flexible body simulations, estimates for required output power are obtained and are listed in Table 9-1.

It is interesting to note that if the CMG system were placed in the hub, rather than in the rotor, the average output power is given by,

$$(P_{\text{out}})_{\text{AV}}^{\text{hub}} = |\omega_T| h \left[P \pm \lambda P \frac{2\alpha}{(\alpha^2 + 1)} \right] \quad (9-11)$$

Where the plus sign is used with "Max I" vehicles, and the minus sign with "Min I" vehicles. Comparing the above equation with equation (9-10) it is seen that:

- o The output power requirement for "Max I" vehicles will always be greater for hub mounted CMG's as compared with rotor mounted CMG's.
- o The output power requirement for "Min I" vehicles will be greater for hub mounted CMG's only if the average gimbal rate is less than 1/2 the spin rate.

Corresponding to the nominal "Y" and "T" configurations, the required output power for a hub mounted CMG would be 0.047 watts and 15.25 watts, respectively. Referring to Table 9-1, it is seen that a rotor mounted CMG requires less output power for both the "T" and "Y" configurations.

PART IIIUSER MANUAL FOR - SPACE STATION GENERAL FLEXIBLE BODY SIMULATIONSTAGE II10.0 Introduction

The digital computer simulation described herein was developed as partial fulfillment of the requirements of Contract NAS 9-11991. The digital program is a special purpose simulation of the flexible body dynamics of the "T" and "Y" space station configurations. The version of the program submitted to MSC, includes the simulation of a Control Moment Gyroscope along with the corresponding control logic used for wobble damping.

Although the digital program is presently assembled to simulate two particular space station configurations, by manipulating the input data, the program can be used to simulate a variety of vehicles with rotating and non-rotating sections, whether it be one rigid body with six degrees of freedom to fifteen flexibly connected bodies with eighty four degrees of freedom. With minor modifications to the program, the simulation can be made to represent dual spin satellites with an arbitrary number of rotor arms (presently limited to three), as well as a high fidelity simulation of a flexible Shuttle Orbiter.

The present version of the program is set up to accept external torques to be applied to any of the fifteen bodies. Thus, torque applying actuators can be placed within any of the bodies. A minor modification to the program is required to allow external forces to be applied to any of the individual bodies.

The digital program is divided into two main sub-programs:

1. The TIME HISTORY PROGRAM (THP) is a numerical integration of the flexible body equations of motion presented in Section 6.0. Wobble damping control is added in the form of a subroutine called CMG (refer to Section 9.2 for the model description). A modal reduction technique is used to decrease the number of coordinates (or integrations) required to describe the vehicle motion.

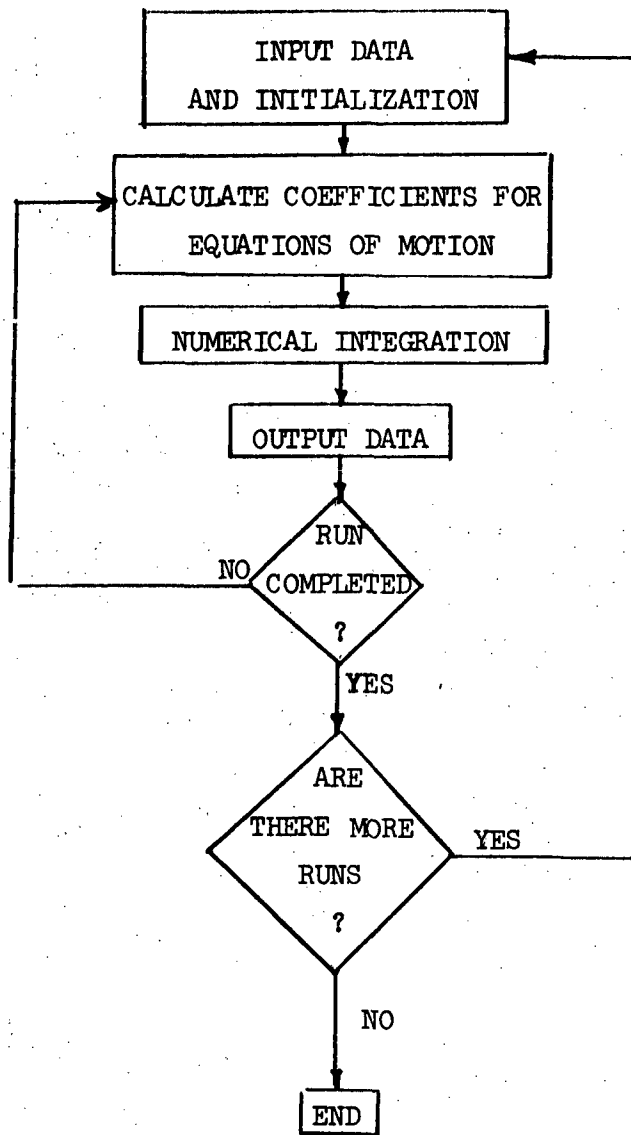
2. The MODAL SERVICE PROGRAM (MSP) is used to determine the mode shapes, frequencies, stiffness matrices, and damping matrices corresponding to the flexible rotor and the section of flexible hub. This data is required for use by the THP. Output from the MSP is in the form of punched cards, in the proper format, for direct input to the THP.

The programming language used was FORTRAN IV. Although the program was developed using the IBM 360/75, particular care was taken to insure compatibility with the UNIVAC 1108 compiler.

A description of the THP and MSP are found in Sections 10.1 and 10.2, respectively.

10.1 TIME HISTORY PROGRAM (THP)

The THP accomplishes the numerical integration of the flexible body equations presented in Section 6.0. The program is structured as follows:



The basic program functions, as pictured above, is accomplished by several subroutines which are described in Section 10.1.2.

The program user can observe at a time increment of his choice, the displacements, rates, and accelerations of each of the 84 coordinates that describe the motion of the vehicle. The number of variables that are actually numerically integrated is equal to the number of flexible modes, chosen by the user, plus eight rigid body modes. The numerical integration technique, known as "Modified Adams," is a corrector-predictor technique that uses a fixed integration interval (DT). A good rule of thumb for choosing a value for DT is that it should be less than (1/20th) the period corresponding to the highest frequency.

The operating procedures for the THP are presented in Section 10.1.3.

10.1.1 Variable Definitions

Table 10.1 contains the definitions of all the program variables used in the Time History Program. The notation used in the equations of motion are used as definitions for program variables that are directly correlated with the equations of motion (refer to Table for definitions).

TABLE 10.1

Definitions of Program Variables

NOTE: The superscripts included in the table below are defined as:

j - lumped hub mass number

k - lumped rotor mass number

i - rotor arm number

Symbol	Definition
AALPHA (I, J, i)	$A (\alpha^i)$
ALPHA (i)	α^i
ANITWO	Logical variable to initialize problem to second antisymmetric mode of Y-rotor
B	CMG gimbal angle
BDE	Desired gimbal angle
BDED, BDEDD	Gimbal rate, gimbal acceleration
BDEF	Filtered desired gimbal angle
EDR	Limited gimbal rate
BETA (I)	β
BIAS	CMG commanded angle bias, filter compensation.
DD(I)	Input data array
DEL x (I), DEL x (I, i), DEL x (I, k, i)	$\Delta_x, \Delta_x^i, \Delta_x^{ik}$
DEL (I), DELD (I)	$\nabla, \dot{\nabla}$
DELTB	δ^*

Symbol	Definition
DT	Iteration time
DTPLOT	Number of seconds per inch of calcomp plot. Also used as logical variable to determine if plotting is desired.
DTPRNT	Print time interval
CH (I, J)	CH
CHG (I, J)	CHG
CONTR	Logical variable to define CMG control or not, CONTR = 0, uncontrolled.
CR (I, J)	CY or CT
ETA	η
FC (I, j)	F_c^j
FD (I, k, i)	F_D^{ik}
FG (I)	F_G
GAM (I), GAMD (I)	$\gamma, \dot{\gamma}$
GPSID (I, J)	$\Gamma(\dot{\phi})$
GYB (I, J)	$\Gamma(Y_B)$
GXL (I, J)	$\Gamma(X_L)$
GYBM (I, J)	(Total rotor mass) • $\Gamma(Y_B)$
GYGB (I, J)	$\Gamma(Y_G - Y_B)$
GYH (I, J)	$\Gamma(Y_H)$
GWR (I, J)	$\Gamma(\omega^r)$
GWX (I, J)	$\Gamma(\omega^X)$
GWXRA (I, J, i)	$\Gamma(\omega^{X^i})$
GWY (I, J)	$\Gamma(\omega^Y)$

Symbol	Definition
GWYC (I, J, i)	$\Gamma(\omega^Y{}^i)$
GWYS (I, J)	$\Gamma^2(\omega^Y)$
H	CMG angular momentum
HCMASS(j)	m_c^j
HMASS	m_A
IA (I, J)	I_A
IB (I, J)	I_B
IC (I, J, j)	I_c^j
ID (I, J, k, i)	I_D^{ik}
IDN	Identification number for data input
IH (I, J)	I_H
IOIN	Data input device indicator
INOUT	Data output device indicator
JT (I, J)	$\pi(\psi) \bullet \pi(\theta)$
KBS1, KBS2	K_{BS1}, K_{BS2}
KBVH (I, J)	KBVH
KBVR (I, J)	KBVR
KH (I, J)	K7H
KHG (I, J)	K6HG
KHS (I, J)	K_{DOCK}
KHV (I, J)	C_{DOCK}
KR (I, J)	KYA or KTA
LAM (I, J)	Equations of motion mass matrix (not reduced)
LMxx (I, J), LMxx (I, J, i), LMxx (I, J, k, i)	$\lambda_{xx}, \lambda_{xx}^i, \lambda_{xx}^{ik}$
LMxB (I, J) LMxxB (I, J, k, i)	$\bar{\lambda}_{xx}, \bar{\lambda}_{xx}^{ik}$

Symbol	Definition
M	Number of physical degrees of freedom
Mx (I, J)	M_x^i, M_x^j, M_x^{ik}
Mx (I, J, i)	
Mx (I, J, k, i)	
MAST	Total vehicle mass
MASS (I, J)	Reduced mass matrix; also used to store inverted mass matrix.
MB (I)	M_B
MBEAR (I)	M_{BEAR}
MC (I, j)	M_c^j
MD (I, k, i)	M_D^{ik}
MDOCK (I)	M_{DOCK}
MG (I)	M_G
N	Number of modal coordinates
NHUB	Number of hub modes
NPOINT	Number of plot points per variable
NROT	Number of rotor modes
NRUN	Run number
NS	Logical variable to determine shuttle attached. NS = 0, shuttle not attached.
NSYM	Number of rotor symmetric modes
NTHET (I, J)	$N(\theta)$
P	Number of rotor arms
PASS	Logical variable defining first pass or not. PASS = 0, first pass; = 1, all other passes.
PBLOCK	Number of plot blocks; three variables plotted per block.
PIBETA (I, J)	$\pi(\beta)$
PIGAM (I, J)	$\pi(\gamma)$
PIPHIC (I, J, j)	$\pi(\phi_c^j)$

Symbol	Definition
PIPHID (I, J, k, i)	$\pi(\phi_{Dik})$
PIPSI (I, J)	$\pi(\psi)$
PITHET (I, J)	$\pi(\theta)$
PHIC (I, j)	ϕ_{Cj}
PHICD (I, j)	$\dot{\phi}_{Cj}$
PHID (I, k, i)	ϕ_{Dik}
PHIDD (I, k, i)	$\dot{\phi}_{Dik}$
P Ψ I (I)	ψ
P Ψ ID (I)	$\dot{\psi}$
PHIMR (I, J)	ϕ_{YRF} or ϕ_{TRF} (rotor mode shapes)
PHIMH (I, J)	ϕ_{HF} (hub mode shapes)
RDMASS (k, i)	m_{Dik}
RH (I)	r_H
RMASS	Total rotor mass
RRMASS	Rotor ring mass
R Γ S (I)	Right side of equations of motion (not reduced)
SGAM (I, J)	$S(\gamma)$
SIX (I), SIX (I, j), SIX (I, k, i)	$\xi_x, \xi_x^j, \xi_x^{ik}$
SIGx (I) SIGx (I, j) SIGx (I, k, i)	$\sigma, \sigma^j, \sigma^{ik}$
SMASS	m_S
SPIN	Nominal rotor spin rate
STHE (I, J)	$S(\theta)$
STHET (I, J)	$S(\theta)^{-1}$

Symbol	Definition
START	CMG start criterion
STOP	CMG stop criterion
SUM1 (I, k, i)	$[x_{Dik} + u_{Dik}]$
T	Simulation time
TAUA (I)	τ_A
TAUC (I, j)	τ_{Cj}
TAUD (I, k, i)	τ_{Dik}
TAUF	CMG filter time constant
TAUL (I)	τ_L
TAUS (I)	τ_S
TC (I)	Control torques applied to vehicle from CMG
TEND	Specified duration of simulation test run (seconds)
THET (I)	θ
THETD (I)	$\dot{\theta}$
RHO (I)	ρ
UC (I, j)	u_{Cj}
UCD (I, j)	\dot{u}_{Cj}
UD (I, k, i)	u_{Dik}
UDD (I, k, i)	\dot{u}_{Dik}
WAIT	Option to delay CMG control until gimbal error is within WAIT radians
WNG	CMG natural frequency
WOKx (I, J)	Work matrices used for intermediate storage
WTMM	Maximum magnitude of the rotor ring transverse rate over one print cycle.
WR (I)	ω^r
WXR (I)	ω^X
WXRA (I, i)	ω^{X^i}
WXY (I)	$\omega^{X/Y}$

Symbol	Definition
WYC (I, j)	ω^y_j
WYH (I)	ω^y
WZ (I, k, i)	ω^z_{ik}
XC (I), XCD (I)	Used as work vectors. Primarily used for storing relative rates and displacements in FLX15
XD (I, k, i)	x^i_{Dk}
XN (I, i)	x^i_N
XL (I)	x_L
XLL (I)	Used as work vector
XVEC (I), XVECD (I), XVECDD (I)	Modal coordinate vector; displacements, rates, and accelerations
YB (I)	y_B
YC (I, j)	y_{Cj}
YE3 (I)	y^3_E
YG (I)	y_G
YH (I)	y_H
ZA (I)	z_A
ZAD (I)	\dot{z}_A
ZETG	CMG damping
ZV3 (I, i)	z^i_3 z_v

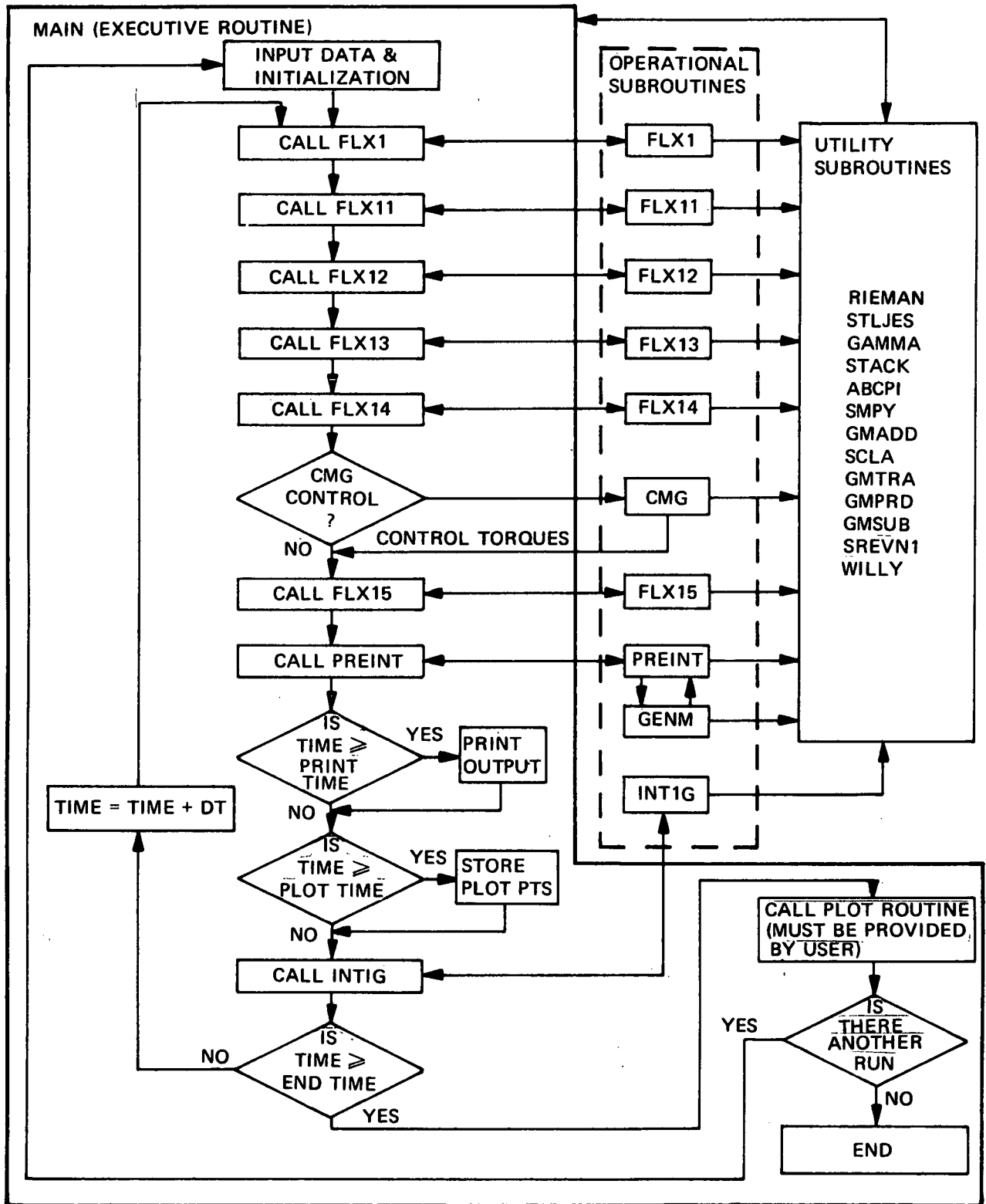


Figure 10-1. Time History Program Flow Diagram

10.1.2 Program Description (with listing)

A flow diagram involving the THP subroutines is shown in Figure 10.1. There are three general subroutine categories; Executive, Operational, and Utility. The MAIN, or Executive, routine controls the sequencing of the operational subroutines, the timing of the iteration process, and the means for inputting and outputting data. The formulation of the equations of motion, and the control system equations, are accomplished within the Operational subroutines. Communication between the MAIN routine and the Operational subroutines is achieved via Common blocks. The Utility subroutines are used by both the MAIN routine and the Operational subroutines to perform special purpose mathematical operations such as: matrix inversion, matrix manipulations (e.g., addition, subtraction, multiplication), vector cross products, coordinate transformations, and numerical integration.

10.1.2.1 MAIN (Executive) Routine

The fortran listing of the routine is shown in Figure 10.2. The program variables are defined in Table 10.1.

The first function of the Main routine is to accept all input data and assign the values to the proper variables. Data format and the required order for the input data is discussed in the Operating Instructions section (Section 10.1.3). Prior to entering the main iteration loop, any variables that require initialization that weren't initialized by input data, are now initialized. During each iteration cycle, the Main routine calls, in sequence, the operational subroutines.

Prior to the integration step (CALL INTIG) tests are made concerning data output. Due to the large number of variables to be observed it was considered convenient to use NAMELIST blocks for printing. A number of the namelists are used for trouble shooting only (e.g., TEST1, TEST2, TEST3). The values of the variables that comprise the main namelist, STATE, are printed every DTPRINT seconds. If a shuttle is attached to the station, or if CMG wobble damping is activated, then the variables in namelists SHUT, and/or CMGPR are also printed every DTPRINT seconds. The important program constants are found in namelist TESTC which is printed at time zero only.

Following the execution of the print instructions, the variables to be plotted are stored on disk to be accessed at the end of the run. It should be noted that the plotting routine provided, PLO3AX, is compatible with the Calcomp system at Grumman and most likely will have to be converted for use at the MSC facility.

If the problem time (T) is less than the specified run duration time (TEND), then the state variables will be integrated via (CALL INTIG) and time will be updated ($T = T + DT$).

10.1.2.2 Operational Subroutines

FLX1, FLX11, FLX12, FLX13, FLX14, FLX15

The coefficients for the flexible body equations are formulated within these subroutines. The next few paragraphs summarize the sequence of operations performed within each FLXxx subroutine. Reference will be made to the actual variables associated with the equations of motion as defined in Section 6.0.

FLX1 (refer to Figure 10.3 for listing)-coordinate transformations are formulated ($A(\gamma)$, $\pi(\beta)$, $\pi(\gamma)$, $\pi(\theta)$, $\pi(\psi)$, $\pi(\phi_{CJ})$, $\pi(\phi_{Dik})$, J). Inertial angular rates are calculated based upon the vehicle state. The vector variable ρ is determined.

FLX11 (Figure 10.4) - Calculate the M matrices and start the formulation of the λ_k matrices.

FLX12 (Figure 10.5) - Complete the formulation of all the λ_k matrices.

FLX13 (Figure 10.6) - Start setting up the right side of the equations of motion by computing Δ_k .

FLX14 (Figure 10.7) - Continue formulating the right side of the equations of motion by computing Δ_k .

FLX15 (Figure 10.8) - Complete the formulation of the right side of the equations of motion. All elastic forces and moments are computed by multiplying the stiffness and damping matrices by the appropriate relative displacements and rates. Finally, σ_k is computed.

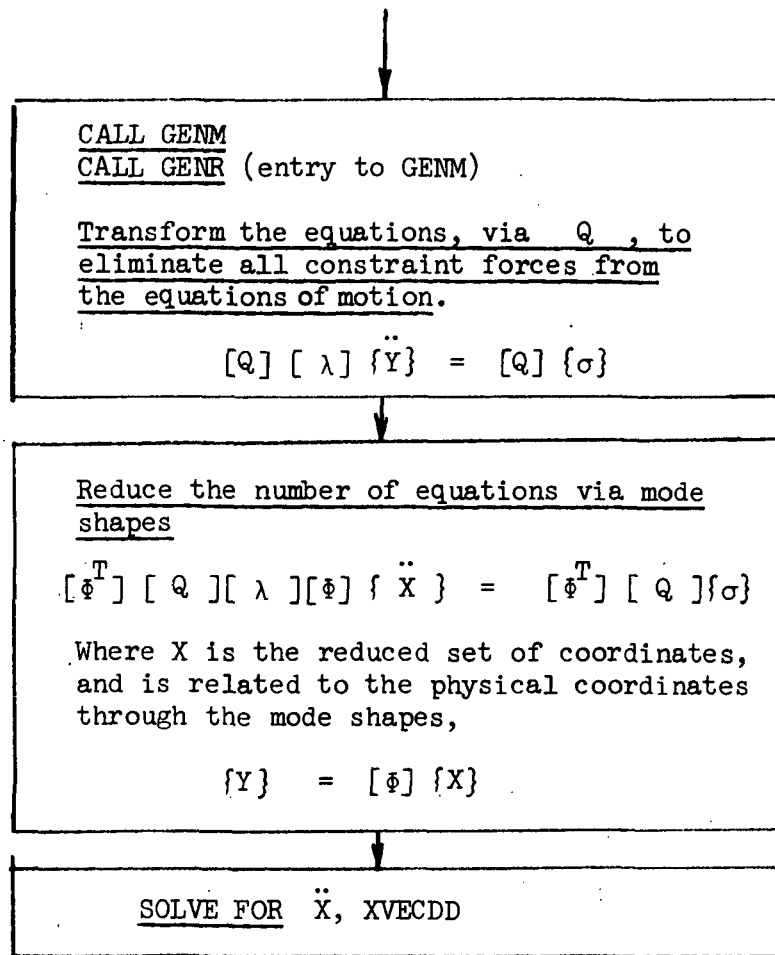
PREINT (Figure 10.9) - The purpose of this subroutine is to solve for the second derivatives of the modal coordinates which are subsequently integrated in subroutine INTIG. The process is outlined below:

Set up total mass matrix λ , and right side σ .

$$[\lambda] \{\ddot{Y}\} = \{\sigma\}$$

Where Y is the physical coordinates, or state vector.





INTIG (Figure 10.10) - Perform a double integration, via CALL RIEMAN, to obtain the displacements and rates in modal coordinates. Transform modal displacements and rates to physical displacements and rates by,

$$\{\dot{Y}\} = [\phi] \{\dot{X}\}, \quad \{Y\} = [\phi] \{X\} \quad (10-1)$$

The modal displacements and rates are initialized within INTIG by applying,

$$\{X(0)\} = (\phi^T \phi)^{-1} \phi^T \{Y(0)\} \quad (10-2)$$

Finally, Euler rates that aren't included in the state vector (e.g. $\dot{\gamma}$, and $\dot{\theta}_1$) are integrated to obtain the corresponding Euler angles.

GENM (Figure 10.11) - Eliminates all constraint forces from the equations of motion.

CMG (Figure 10.12) - Develops control torques, TC, for wobble damping. The simulated CMG configuration and associated control logic is presented in Section 9.2.

The user has the option to place the CMG in any of the bodies that form the flexible vehicle. This is accomplished by the first and last three executable statements in the subroutine. The first three statements equate the angular rates of the body to the rates

used within the control equations. The last three statements equate the external torques applied to the particular body to the derived control torques.

Control variables that require numerical integration is accomplished via a direct call to STLJES from CMG.

It is noted that an additional common block, called CONT, is used to communicate between MAIN and CMG.

10.1.2.3 Utility Subroutines

RIEMAN (Figure 10.13) - A Modified-Adams numerical integration, for a second order differential equation, using a fixed integration interval. Basically, the integrated variable at $t + \Delta t$ is approximated by the first three terms of a Taylor series expansion about Δt . That is,

$$f(t + \Delta t) = f(t) + \dot{f}(t) \Delta t + \ddot{f}(t) \frac{\Delta t^2}{2} \quad (10-3)$$

The integration scheme is a corrector-predictor technique where the predicted value (i.e., calculated at t for $t + \Delta t$) is corrected during the next iteration by applying current rates and accelerations to the above expansion.

STLJES (Figure 10.14) - A special version of RIEMAN for the purpose of integrating a first order differential equation.

GAMMA (Figure 10.15) - The GAMMA matrix, $\Gamma(a)$, is formed to facilitate the vector cross product operation. Specifically,

$$\{a\} \times \{b\} = \Gamma(a)\{b\} \quad (10-4)$$

STACK (Figure 10.16) - Places small matrices at specified locations within a large matrix.

ABCPI (Figure 10.17) - Formulates the coordinate transformation $(\pi(\theta))$ for three ordered rotations ($\theta_1, \theta_2, \theta_3$), and/or the coordinate transformation for a single rotation $A(\theta_1)$.

SMPY (Figure 10.18) - Multiplies a scalar by a matrix. (Part of IBM Scientific Subroutine Package).

GMADD (Figure 10.19) - Matrix Addition (IBM Scientific Subroutine Package).

SCLA (Figure 10.20) - Set each element of a matrix equal to a given scalar. (IBM Scientific Subroutine Package).

GMTRA (Figure 10.21) - Matrix transpose. (Part of IBM Scientific Subroutine Package)

GMFRD (Figure 10.22) - Multiplies two matrices (Part of IBM Scientific Subroutine Package)

GMSUB (Figure 10.23) - Matrix subtraction. (Part of IBM Scientific Subroutine Package).

SREVN1 (Figure 10.24) - Inverts a real, single precision matrix. The method used is Gauss-Jordan elimination with partial pivoting.

WILLY (Figure 10.25) - Prints a matrix in an easily readable format.

20.1 (AUG 71)

OS/360 FORTRAN H

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COMPILER OPTIONS - NAME= MAIN,GET=02,LINECNT=60,SIZE=0000K,
SOURCE,PCD,NOLIST,DECK,LOAD,MAP,NOEDIT,LD,NDXREF
0002   IMPLICIT REAL (A-Z)
0003   COMMON/TIME/T,DT,NSYM,ANTTWC
0004   COMMON/CONT/
*TC(3),TAUF,START,STOP,H,WNG,ZETG,BDR,B,FE,BDE,BDEF,BDEDD,BDED,
*BIAS,WAIT
0005   COMMON/FLX/
*ALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
*DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
*CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
*EIH(3),EIP(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH,
*GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
*GFH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
*GWYS(3,3),GYR(3,3),GYRM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0006   COMMON/FLX/
*IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KRS1,KRS2,
*KBVH(3,3),KBVR(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
*LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
*LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
*LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
*LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
*LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),KHG(6,18),
*LM16B(3,3,3,3),KP(57,57),KH(18,18)
0007   COMMON/FLX/
*M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
*MR(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
*N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
*PIPSI(3,3),PIPHET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
*PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0008   COMMON/FLX/
*PDMASS(3,3),PH(3),PJ,RJ,PMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
*SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
*SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
*SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
*THET(3),THETD(3),FHC(3),STHE(3,3),SPIN
0009   COMMON/FLX/
*UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
*WOK3(84),WOK4(3,3),WOK5(3,3),WCK5(3,3),WR(3),WKR(3),WXRA(3,3),
*WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
*XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
*YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0010   NAMELIST/TIME/T
0011   NAMELIST/STATE/ZAD,ZA,WYH,GAM,UCD,UC,PHICD,PHIC,DELD,DEL,WXR,
*UDD,UD,PHIDD,PHID,WZ,XVEC,XVECD,XVECD,RTS,WGMM
0012   NAMELIST/SHLT/PSID,PSI,WR,JT,MDOCK,PIPSI,TAUS
0013   NAMELIST/TESTC/ALPHA,BETA,ETA,HMASS,HCMASS,IA,IB,IC,ID,IH,
*KRS1,KRS2,KBVH,KBVR,KHS,KHV,M,MAST,NROT,NHUB,N,NS,P,PASS,
*GXL,GPH,GYR,GYRM,GYH,PIBETA,PDMASS,RH,RMASS,RRMASS,SMASS,SPIN,
*XN,XL,YR,YE3,YG,YH,ZV3,CP,CH,CHG,KHG,KR,KH,PHIMR,PHIMH
0014   NAMELIST/TEST1/DELI,DEL2,DEL3,DEL4,DEL5,DEL6,DEL7,DEL8,FC,FD,FG,
*GAMD,GYGB,GWXRA,GWY,GWYS,LM1,LM2,LM3,LM4,LM5,
*LM6,LM7,LM8,LM9,LM10,LM11,LM12,LM13,LM14,LM15,LM16,LM17,LM18,LM19,
*LM20,LM21,LM22,LM23,LM24,LM25,LM26,LM4B,LM13B,LM16B
0015   NAMELIST/TEST2/M1,M2,M3,M4,M5,M8,MBEAR,MC,MD,MG,PIGAM,
*PIPHIC,PIPHID,PIPHET,NTHET,SI1,SI2,SI3,SI4,SIG1,SIG2,SIG3,
*SIG4,SIG5,SIG6,SIG7,SIG8,SGAM,STHET,TAJA,TAUC,TAJD,TAUL,THET,
*THETD,PHO,STHE,WXRA,WXY,WYC,WZ,XD,YC

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0016  NAMELIST/TEST3/RTS,XVEC,XVECD,XVECDD
0017  NAMELIST/CMGRP/WXP,FE,PDF,BDEF,R,BDEDD,ROED,ROR,FC
0018  INTEGER II,JJ,KK,LL,NN
0019  INTEGER IOIN,IOIN,IOIN,NRUN
0020  INTEGER NPOT,NHUB,N,M,NFCINT,NSYM
0021  INTEGER NROW(5),CCL
0022  COMMON/BUF/BUFFER(512)
0023  DIMENSION VALU(5)
0024  DIMENSION QNAM(11)
0025  IOIN=5
0026  IOIN=6
0027  PAUSED=0.0
0028  DO 1 II=1,500
0029    1 DD(II)=0.0
0030  121 READ(IOIN,22)NRUN,QNAM
0031    22 FORMAT(I3,11A4)
0032    WRITE(IOIN,23)NRUN,QNAM
0033    23 FORMAT(1H1,'RUN NUMBER=',I3,5X,11A4/)
0034    IF(NRUN.GT.0)GO TO 25
0035    IF(PAUSED.EQ.0.0)GO TO 24
0036    CALL EPLOTT
0037  24 CALL EXIT
0038  25 READ(IOIN,26)IDN,VAL,QNAM
0039    26 FORMAT(I5,E20.5,11A4)
0040    IF(IDN.LE.0.0)GO TO 27
0041    DD(IDN)=VAL
0042    WRITE(IOIN,29)IDN,VAL,QNAM
0043    29 FORMAT(5X,'DD(',I5,')=',E20.5,5X,11A4)
0044    GO TO 25
0045  27 DTPLOT=DD(1)
0046    IF(DTPLOT.LE.0.0.CR.PAUSED.NE.0.0)GO TO 75
0047    CALL PLOTS(BUFFER,512)
0048    PAUSED=1.0
0049  75 CONTINUE
0050  C INPUT DATA ARRAY
0051  C ITERATION TIME
0052  C RUN TIME
0053  C TEND=DD(3)
0054  C PRINT INTERVAL
0055  C DTPRNT=DD(4)
0056  C MASS PROPERTY DATA
0057  C RIGID HUB MASS
0058  C HMASS=DD(7)/32.174
0059  C SHUTTLE MASS - SET TO ZERO FOR NO SHUTTLE
0060  C SMASS=DD(8)/32.174
0061  C SHUTTLE ATTACHED NS=1.,NOT ATTACHED NS=0.
0062  C NS=DD(5)
0063  C NUMBER OF ROTOR ARMS
0064  C P=DD(6)
0065  C ROTOR RING MASS
0066  C PRMASS=DD(9)/72.174
0067  C DO 10 II=1,3
0068  C MASS OF LUMPED HUB MASS
0069  C HCMASS(II)=DD(9+II)/32.174
0070  C MASS OF LUMPED ROTOR MASS-EACH ARM IS SAME
0071  C RDMASS(II,1)=DD(12+II)/32.174
0072  C RDMASS(II,2)=RDMASS(II,1)

```

GAC PLOT ROUTINES.
 REPLACE BY EQUIVALENT
 MSC PLOT ROUTINES

```

0066     IF(P.EQ.2.)GO TO 8
0068     RDMASS(II,3)=RDMASS(II,2)
0069     GO TO 10
0070     P RDMASS(II,3)=0.0
0071 10 CONTINUE
0072     DO 15 II=1,3
0073     DO 15 JJ=1,3
C       RIGID HUB INERTIA
0074     NN=15+JJ+3*(II-1)
0075     IA(JJ,II)=DD(NN)
C       SHUTTLE MASS - INPUT ZERO FOR NO SHUTTLE
0076     IH(JJ,II)=DD(NN+9)
C       ROTOR RING INERTIA
0077     IR(JJ,II)=DD(NN+18)
0078     DO 15 KK=1,3
C       INERTIA OF LUMPED HUB MASS - II REPRESENTS EACH LUMPED MASS
0079     LL=42+KK+3*(JJ-1)+9*(II-1)
0080     IC(KK,JJ,II)=DD(LL)
C       INERTIA OF LUMPED ROTOR MASS-ALL ARMS IDENTICAL
0081     ID(KK,JJ,II,1)=DD(LL+27)
0082     ID(KK,JJ,II,2)=ID(KK,JJ,II,1)
0083     IF(P.EQ.2.)GO TO 12
0085     ID(KK,JJ,II,3)=ID(KK,JJ,II,2)
0086     GO TO 15
0087 12 ID(KK,JJ,II,3)=0.0
0088 15 CONTINUE
C       GEOMETRY
0089     DO 30 II=1,3
C       LOCATE SHUTTLE
0090     YH(II)=DD(96+II)
0091     RH(II)=DD(99+II)
C       LOCATE ROTOR RING PIVOT
0092     YR(II)=DD(102+II)
C       LOCATE FLEX HUB
0093     YG(II)=DD(105+II)
0094     YF3(II)=DD(108+II)
C       LOCATE MASS CENTER OF ROTOR RING
0095     XL(II)=DD(111+II)
C       LOCATE FLEX ARM - ALL ARMS IDENTICAL
0096     XN(II,1)=DD(114+II)
0097     XN(II,2)=XN(II,1)
0098     XN(II,3)=XN(II,2)
0099     ZV3(II,1)=DD(117+II)
0100     ZV3(II,2)=ZV3(II,1)
0101     ZV3(II,3)=ZV3(II,2)
0102     DO 30 JJ=1,3
0103     NN=120+JJ+3*(II-1)
C       LOCATE HUB LUMPED MASS
0104     YC(JJ,II)=DD(NN+90)
C       LOCATE ROTOR LUMPED MASS
0105     XD(JJ,II,1)=DD(NN)
0106     XD(JJ,II,2)=DD(NN)
0107 20 XD(JJ,II,3)=DD(NN)
C       ARM ROTATION
0108     ALPHA(1)=0.
0109     ALPHA(2)=DD(130)
0110     ALPHA(3)=DD(131)
C       SHUTTLE ROTATION

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```

0111      DD 40 II=1,3
0112      4C BETA(II)=DD(131+II)
C STIFFNESS AND VISCOSITIES
C BEARING
0113      KRS1=DD(135)
0114      KRS2=DD(136)
C DISCONTINUITY DEFLECTION
0115      DEL TR=DD(137)
C AMOUNT OF DAMPING ON INNER RACE (ZERO TO ONE)
0116      ETA=DD(138)
0117      DD 50 II=1,3
0118      DD 50 JJ=1,3
C OUTER RACE DAMPING
0119      NN=JJ+3*(II-1)+138
0120      KRVP(JJ,II)=DD(NN)
C INNER RACE DAMPING
0121      KRVP(JJ,II)=DD(NN+9)
C SHUTTLE PIVOT STIFFNESS
0122      KHS(JJ,II)=DD(NN+18)
C SHUTTLE PIVOT DAMPING
0123      50 KHV(JJ,II)=DD(NN+27)
C ROTOR ARMS - ALL IDENTICAL
C E
0124      ER=DD(175)
C I
0125      RI=DD(176)
C G
0126      GR=DD(177)
C J
0127      RJ=DD(178)
C HUB SECTIONS
C E
0128      EH=DD(179)
C I
0129      HI=DD(180)
C G
0130      GH=DD(181)
C J
0131      HJ=DD(182)
C FLOATING EACH SECTION
0132      EIP(1)=ER*RI
0133      GJP(1)=GR*RJ
0134      EIH(1)=EH*HI
0135      GJH(1)=GH*HJ
0136      DD 60 II=2,3,1
0137      EIP(II)=EIP(1)
0138      GJP(II)=GJP(1)
0139      EIH(II)=EIH(1)
0140      50 GJH(II)=GJH(1)
C NOMINAL SPIN RATE
0141      SPIN=DD(200)
0142      IF(NR/LN.EQ.1)GO TO 315
0144      IF(NPCT.EQ.DD(210))GO TO 315
C ZERO PHIMP FOR NEW MODAL DATA
0146      DD 310 II=1,57
0147      DD 310 JJ=1,57
0148      310 PHIMP(JJ,II)=0.0
0149      315 CONTINUE

```

```

C NUMBER OF ROTOR MODES
0150 NROT=DD(210)
C NUMBER OF HUB MODES
0151 NHUB=DD(211)
C NUMBER OF SYM. ROTCF MODES
0152 NSYM=DD(212)
C ANTTWO=1. TO INITIALIAZE THE RCTOR 2ND ANTI-SYM MODE
0153 ANTTWO=DD(213)
C NUMBER OF PLOT BLOCKS
0154 PBLOCK=DD(214)
C DATA FOR CMG CONTROL
C FOR CMG CONTROL SET CONT .GT. 0.0
0155 CONTR=DD(220)
C STOP DAMPING FOR WOBBLE .LT. STOP
0156 STOP=DD(221)
C START DAMPING FOR WOBBLE .GT. START
0157 START=DD(222)
C TIME CONSTANT OF GIMBAL ANGLE CCMAND LOW PASS FILTER,NO FILTER IF
0158 TAUF=DD(223)
C ANGULAR MOMENTUM OF WHEEL
0159 H=DD(224)
C NATURAL FREQ. OF TORQUE MOTOR
0160 WNG=DD(225)
C DAMPING OF TORQUE MOTOR
0161 ZETG=DD(226)
C BIAS=+OR- WOBBLE FREQ. MINUS FOR MINI
C NOTE.. SET BIAS=0. FOR NO GIMBAL ANGLE BIAS
0162 BIAS=DD(227)
C WAIT OPTION FOR ACTIVE CONTROL
C SET WAIT=MIN. ERROP(IN RAD)BEFORE ACTIVE CONTR IS ALLOWED
0163 WAIT=DD(228)
C INITIALIZE STATE VECTOR
0164 DO 70 II=1,3
0165 ZA(II)=DD(300+II)
0166 PSI(II)=DD(306+II)
0167 DO 70 JJ=1,3
0168 NN=9+JJ+3*(II-1)
0169 UC(JJ,II)=DD(NN+300)
0170 PHIC(JJ,II)=DD(NN+309)
0171 DO 70 KK=1,3
0172 LL=30+KK+3*(JJ-1)+18*(II-1)
0173 UD(KK,JJ,II)=DD(LL+300)
0174 70 PHID(KK,JJ,II)=DD(LL+309)
C INITIALIZE DERIVATIVE OF STATE
0175 DO 80 II=1,3
0176 ZAD(II)=DD(400+II)
0177 WYH(II)=DD(403+II)
0178 PSID(II)=DD(406+II)
0179 WXY(II)=DD(427+II)
0180 DO 80 JJ=1,3
0181 NN=9+JJ+3*(II-1)
0182 UCD(JJ,II)=DD(NN+400)
0183 PHICD(JJ,II)=DD(NN+409)
0184 DO 80 KK=1,3
0185 LL=30+KK+3*(JJ-1)+18*(II-1)
0186 UDD(KK,JJ,II)=DD(LL+400)
0187 80 PHIDD(KK,JJ,II)=DD(LL+409)
C INITIALIZE EULER ANGLES

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0188      DD 90 II=1,3
0189      GAM(II)=DD(484+II)
0190      90 THET(II)=DD(487+II)
C
C  ZERO MODAL STIFFNESS AND DAMPING MATRICES
0191      IF(NPUN.GT.1)GO TO 205
0193      DO 201 JJ=1,57
0194      DO 201 II=1,57
0195      PHIMR(II,JJ)=0.0
0196      KR(II,JJ)=0.0
0197      201 CR(II,JJ)=0.0
0198      DO 204 JJ=1,18
0199      DO 203 II=1,18
0200      PHIMH(II,JJ)=0.0
0201      KH(II,JJ)=0.0
0202      203 CH(II,JJ)=0.0
0203      DO 204 II=1,6
0204      KHG(II,JJ)=0.0
0205      204 CHG(II,JJ)=0.0
0206      205 CONTINUE
C  INPUT MODAL STIFFNESS AND DAMPING MATRICES
0207      206 READ(10IN,207)CCL,(NRCW(JJ),VALU(JJ),JJ=1,5),IDN
0208      207 FORMAT(I3,5(I3,F12.5),I2)
0209      IF(IDN.EQ.0)GO TO 300
0211      II=1
0212      IF(IDN.GT.1)GO TO 208
0214      LL=12
0215      IF(P.EQ.3.)LL=28
0217      IF(COL.GT.LL)GO TO 209
0219      IF(COL.GT.NSYM)GO TO 206
0221      GO TO 208
0222      209 COL=CCL-LL+NSYM
0223      IF(COL.GT.NROT)GO TO 206
0225      208 IF(NRCW(II).EQ.0)GO TO 206
0227      GO TO (210,220,230,240,250,260,270,290),IDN
C
C          IDN          INPUT DATA TO-
C          1          FHIMF
C          2          PHIMH
C          3          KTA CR KYA
C          4          CT CR CY
C          5          KH
C          6          CH
C          7          KHG
C          8          CHG
C  READ IN MODAL DATA - ERROR
0228      210 PHIMR(NROW(II),CCL)=VALU(II)
0229      GO TO 290
C  HUB MODES
0230      220 PHIMH(NROW(II),CCL)=VALU(II)
0231      GO TO 290
C  INPUT ROTOR STIFFNESS AND DAMPING MATRICES
C  STIFFNESS
0232      230 KR(NROW(II),COL)=VALU(II)
0233      GO TO 290
C  DAMPING
0234      240 CR(NROW(II),COL)=VALU(II)
0235      GO TO 290
C  INPUT HUB STIFFNESS AND DAMPING

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```

C STIFFNESS
0236 250 KH(NROW(II),COL)=VALU(II)
0237 GO TO 290
C DAMPING
0238 260 CH(NROW(II),COL)=VALU(II)
0239 GO TO 290
C STIFFNESS FOR FORCES AT POINT G
0240 270 KHG(NROW(II),COL)=VALU(II)
0241 GO TO 290
C DAMPING FOR FORCES AT POINT G
0242 280 CHG(NROW(II),COL)=VALU(II)
0243 290 II=II+1
0244 IF(II.LT.6)GO TO 208
0246 GO TO 206
0247 300 CONTINUE
C INITIALIZATION BLOCK
C TIME
0248 T=0.
0249 PASS=C.
0250 WTMM=C.0
C PRINT AND PLOT INTERVALS
0251 IF(DTPLT.GT.0.)DTPL=TEAD/850.
0253 TPRINT=0.0
0254 TPLDT=0.0
0255 NPRINT=0
C GIMBAL ANGLE AND GIMBAL RATE
0256 B=0.0
0257 BDP=0.0
C EXTERNAL TORQUES
0258 DO 460 II=1,3
0259 TAU1(II)=0.0
0260 TAU2(II)=0.0
0261 TAU3(II)=0.0
0262 DO 460 JJ=1,3
0263 TAU4(JJ,II)=0.0
0264 DO 460 KK=1,3
0265 460 TAU5(KK,JJ,II)=0.0
C
C MAIN ITERATION LOOP
0266 150 CALL FLX1
0267 CALL FLX11
0268 CALL FLX12
0269 CALL FLX13
0270 CALL FLX14
0271 IF(CONTR.EQ.0.)GO TO 151
0273 CALL CMG
0274 151 CONTINUE
0275 CALL FLX15
C CALCULATE WOBBLE MAGNITUDE
0276 WTM=SQRT(WXR(2)**2+WXR(3)**2)
C MAX WOBBLE OVER PRINT CYCLE
0277 WTMM=AMAX1(WTM,WTMM)
0278 IF(T.LT.TPRINT)GO TO 152
0280 WRITE(IOUT,TIM)
0281 152 CALL PREINT
0282 IF(T.LT.TPRINT)GO TO 155
0284 TPRINT=TPRINT+DTPRNT
0285 WRITE(IOUT,STATE)

```



```

0286      WTMM=C.0
0287      IF(CONTR.EQ.0.)GO TO 156
0289      WRITE (IOUT,CMGPR)
0290      156 CONTINUE
0291      IF(T.GT.-.03)GO TO 153
0293      CALL WILLY(' LAM ',LAM,84,94,1,94,1,84)
C      CHECK OUTPUT
0294      WRITE (IOUT,TEST1)
0295      WRITE (IOUT,TEST2)
0296      WRITE (IOUT,TEST3)
0297      153 IF(PASS.GT.C.)GO TO 154
0299      WRITE (IOUT,TESTC)
0300      154 IF(NS.EQ.0.)GO TO 155
0302      WRITE (IOUT,SHUT)
0303      155 CONTINUE
0304      IF(DTPLT.LE.0.)GO TO 125
0306      IF(PASS.GT.C.)GO TO 122
0308      WRITE (1)NRUN,PBLOCK,TEND,DTPLT
0309      122 CONTINUE
0310      IF(T.LT.TPLOT)GO TO 125
0312      TPLOT=TPLOT+DTPL
0313      IF(NPOINT.GE.850)GO TO 125
0315      NPOINT=NPOINT+1
0316      WRITE (1)T,WXR(2),WXR(3),WYH(2)
0317      WRITE (1)WYH(3),WYC(3,3),UCD(2,3)
0318      WRITE (1)PSID(1),UDD(1,3,2),UDD(2,3,2)
0319      WRITE (1)WZ(2,3,1),RDR,R
0320      125 CONTINUE
0321      IF(T+DT.LT.TEND)GO TO 120
0323      IF(DTPLT.LE.0.)GO TO 121
0325      END FILE 1
0326      CALL PLO3AX
0327      GO TO 121
0328      120 CONTINUE
0329      CALL INTIG
0330      T=T+DT
0331      PASS=1.
0332      GO TO 150
0333      END

```

20.1 (AUG 71)

05/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

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0002      SUBROUTINE FLX1
0007      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DELS(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *FIH(3),FIP(3),FH,FP,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH,
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGR(3,3),GXL(3,3),DD(500),
          *GRH(3,3),GWP(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYB(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KBVP(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),KHG(6,18),
          *LM16B(3,3,3,3),KP(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MB(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PI THET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *PDMASS(3,3),PH(3),PI,RJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(3),THE TD(3),RHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WOKS(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),Y C(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
0010      DIMENSION DUMMY(3,3)
          C SET UP TRANSFORMATION MATRICES
          C TRANSFORMATION A OF ALPHA-STEP1
0011      IF(PASS.GT.C)GO TO 10
0013      DO 2 II=1,3
0014      DO 1 JJ=1,3
0015          1 AALPHA(JJ,II,1)=0.0
0016          2 AALPHA(II,II,1)=1.0
0017      KK=P
0018      DO 5 II=2,KK,1
0019          5 CALL ARCP1(AALPHA(1,1,II),DUMMY,ALPHA(II),0)
          C ADJUST AALPHA FOR NUMERICAL INACCURACIES
0020      IF(P.EQ.3)GO TO 6
0022      AALPHA(2,2,2)=-1.0
0023      AALPHA(3,2,2)=0.0
0024      AALPHA(2,3,2)=0.0
0025      AALPHA(3,3,2)=-1.0
0026      GO TO 7

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0027      6 AALPHA(2,2,2)=-.50
0028      AALPHA(3,2,2)=-.8660254
0029      AALPHA(2,3,2)=+.8660254
0030      AALPHA(3,3,2)=-.50
0031      AALPHA(2,2,3)=-.50
0032      AALPHA(3,2,3)=+.8660254
0033      AALPHA(2,3,3)=-.8660254
0034      AALPHA(3,3,3)=-.50
0035      7 CONTINUE
C      TRANSFORMATION PI OF BETA - STEPS 2 AND 3
0036      CALL ARCP1(DUMMY,PIBETA,BETA,1)
0037      10 CONTINUE
C      TRANSFORMATIONS PI OF GAMMA,THETA,PSI,PHI OF C AND D.-STEPS 5,6
0038      CALL ARCP1(DUMMY,PIGAM,GAM,1)
0039      CALL ARCP1(DUMMY,PI THET,THET,1)
0040      CALL ARCP1(DUMMY,PI PSI,PSI,1)
0041      DO 20 II=1,3
0042      CALL ABCPI(DUMMY,PIPHIC(1,1,II),PHIC(1,II),1)
0043      DO 20 JJ=1,3
0044      20 CALL ABCPI(DUMMY,PIPHID(1,1,JJ,II),PHID(1,JJ,II),1)
C      TRANSFORMATION J -FROM HUP TO SHUTTLE - STEP 7
0045      CALL GMPRO(PI PSI,PIBETA,JT,3,3,3)
C
C      SET UP FOR EULER RATE CALCULATIONS - STEP 8
0046      S2=SIN(GAM(2))
0047      S3=SIN(GAM(3))
0048      C2=COS(GAM(2))
0049      C3=COS(GAM(3))
0050      SGAM(1,1)=C3/C2
0051      SGAM(2,1)=S3
0052      SGAM(3,1)=-C3*S2/C2
0053      SGAM(1,2)=-S3/C2
0054      SGAM(2,2)=C3
0055      SGAM(3,2)=S3*S2/C2
0056      SGAM(1,3)=0.
0057      SGAM(2,3)=0.
0058      SGAM(3,3)=1.
0059      S2=SIN(THET(2))
0060      S3=SIN(THET(3))
0061      C2=COS(THET(2))
0062      C3=COS(THET(3))
0063      STHET(1,1)=C3/C2
0064      STHET(2,1)=S3
0065      STHET(3,1)=-C3*S2/C2
0066      STHET(1,2)=-S3/C2
0067      STHET(2,2)=C3
0068      STHET(3,2)=S3*S2/C2
0069      STHET(1,3)=0.
0070      STHET(2,3)=0.
0071      STHET(3,3)=1.
C      COMPUTE STHET INVERSE
0072      STHE(1,1)=C3*C2
0073      STHE(2,1)=-S3*C2
0074      STHE(3,1)=S2
0075      STHE(1,2)=S3
0076      STHE(2,2)=C3
0077      STHE(3,2)=0.
0078      STHE(1,3)=0.

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0079     STHE(2,3)=0.
0080     STHE(2,3)=1.
C   CALCULATE NTHET
0081     NTHET(1,1)=1.-STHE(1,1)
0082     NTHET(2,1)=-STHE(2,1)
0083     NTHET(3,1)=-STHE(3,1)
0084     NTHET(1,2)=0.0
0085     NTHET(2,2)=1.0
0086     NTHET(3,2)=0.0
0087     NTHET(1,3)=0.0
0088     NTHET(2,3)=0.0
0089     NTHET(3,3)=1.0

C
C   COMPUTE INERTIAL ANGULAR RATES-STEP 9 ,WXR,WXY-ROTOR RING
0090     CALL GMPRD(PI THET,WYH,WCK1,3,3,1)
0091     IF(PASS.EQ.C.)GO TO 30
0093     CALL GMPRD(STHE,THETD,WXY,3,3,1)
0094     30 CALL GMADD(WXY,WCK1,WXR,3,1)

C
0095     KK=P
C   WR - SHUTTLE RATES
0096     35 IF(NS.EQ.C.)GO TO 40
0098     CALL GMPRD(JT,WYH,WCK1,3,3,1)
0099     CALL GMADD(WOK1,PSID,WR,3,1)
0100     40 DO 50 II=1,3
C   WYC - RATES OF HUB LUMPED MASSES
0101     CALL GMPRD(PIPHIC(1,1,II),WYH,WOK1,3,3,1)
0102     CALL GMADD(WOK1,PHICD(1,II),WYC(1,II),3,1)
C   WXRA - RATES OF ROTOR ARMS
0103     IF(II.GT.KK)GO TO 50
0105     CALL GMPRD(AALPHA(1,1,II),WXR,WXRA(1,II),3,3,1)
0106     50 CONTINUE
C   WZ - RATES OF ROTOR LUMPED MASSES
0107     DO 55 JJ=1,KK
0108     DO 55 II=1,3
0109     CALL GMPRD(PIPHID(1,1,II,JJ),WXRA(1,JJ),WOK1,3,3,1)
0110     CALL GMADD(WOK1,PHIDD(1,II,JJ),WZ(1,II,JJ),3,1)
0111     55 CONTINUE

C
C   EULER RATES FIRST PASS ONLY
0112     CALL GMPRD(PI THET,WYH,WCK1,3,3,1)
0113     IF(PASS.GT.0.)GO TO 70
C   GAMMA-DOT
0115     CALL GMPRD(SGAM,WYH,GAMD,3,3,1)
C   THETA-DOT
0116     CALL GMPRD(STHET,WXY,THETD,3,3,1)
C   CALCULATE INITIAL VALUES FOR DEL AND DELD
0117     DEL(1)=0
0118     DEL(2)=THET(2)
0119     DEL(3)=THET(3)
0120     DELD(1)=THE TD(1)-SPIN+WOK1(1,1)
0121     DELD(2)=THE TD(2)
0122     DELD(3)=THE TD(3)
0123     70 CONTINUE
C   CALCULATE PHO
0124     CALL GAMMA(WOK4,WXY)
0125     CALL GMPRD(WOK4,WCK1,WOK5,3,3,1)
0126     CALL GMPRD(NTHET,WCK5,WCK1,3,3,1)

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0127 ..... RHO(1)=THE TD(3)*THE TD(2)-WCK1(1,1) .....  
0128 ..... RHO(2)=-THE TD(3)*THE TD(1)-WCK1(2,1) .....  
0129 ..... RHO(3)=THE TD(2)*THE TD(1)-WCK1(3,1) .....  
.....  
0130 ..... RETURN .....  
0131 ..... END .....
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20.1 (AUG 71)

CS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

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0002      SUBROUTINE FLX11
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DELS(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *FJH(3),FIP(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH,
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GPH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYB(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(3,3),IR(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KRVP(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),KHG(6,18),
          *LM16B(3,3,3,3),KR(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MB(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBF1A(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PI THE T(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *RDMASS(3,3),RH(3),RI,RJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(3),THEID(3),FHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WCK1(3,3),WCK2(84,84),
          *WCK3(24),WCK4(3,3),WCK5(3,3),WCK6(3,3),WR(3),WKR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
0010      DIMENSION W7(3,3)
          C COMPUTE THE M MATRICES THAT WILL BE USED AS PART OF THE MASS MATRIX
          C FORMULATION - STEP 11      M1
0011      IF(PASS.GT.C.)GO TO 10
0013      CALL GAMMA(GXL,XL)
0014      10 CALL GMTRA(FITHE T,WCK1,3,3)
0015      CALL GMPRD(WCK1,GXL,M1,3,3,3)
0016      CALL SMPY(M1,-1.,M1,3,3)
          C M4,M5
0017      IF(PASS.GT.C.)GO TO 20
0019      CALL GAMMA(GPH,PH)
0020      CALL GAMMA(GYH,YH)
0021      20 CALL GMTRA(JT,WCK1,3,3)
0022      CALL GMPRD(WCK1,GPH,M5,3,3,3)
0023      CALL GMPRD(M5,JT,WCK1,3,3,3)
0024      CALL GMSUB(WCK1,GYH,M4,3,3)
          C M2,M3
0025      KK=P

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0026      DO 30 II=1, KK
0027      CALL GMPRD(AALPHA(1,1,II), PITHET, WOK1, 3, 3, 3)
0028      CALL GMTRA(WOK1, M2(1,1,II), 3, 3)
0029      DO 30 JJ=1, 3
0030      CALL GMADD(XD(1, JJ, II), UD(1, JJ, II), SUM1(1, JJ, II), 3, 1)
0031      CALL GAMMA(WOK1, SUM1(1, JJ, II))
0032      CALL GMPRD(WOK1, AALPHA(1,1,II), WOK4, 3, 3, 3)
0033      CALL GMPRD(M2(1,1,II), WCK4, WCK3, 3, 3, 3)
0034      CALL SMPY(WCK3, -1., M3(1,1, JJ, II), 3, 3)
0035      30 CONTINUE

```

C

C

C CALCULATE THE LAMDA MATRICES

C

COMBINE SOME MASSES

```

0036      IF(PASS.GT.C.) GO TO 60
0038      RMASS=0.
0039      DO 40 II=1, KK
0040      DO 40 JJ=1, 3
0041      40 RMASS=RMASS(PDMASS(JJ, II))+RMASS
0042      RMASS=RMASS+RMASS
0043      MAST=C.
0044      DO 45 II=1, 3
0045      45 MAST=MAST+HCMASS(II)
0046      MAST=MAST+RMASS+HMASS+SMASS

```

C

C SOME CONSTANT LAMDA MATRICES FOLLOW -LM3, LM9, LM15, LM17, LM20, LM12

```

0047      DO 55 II=1, 3
0048      DO 55 JJ=1, 3
0049      DO 50 KK=1, 3
0050      LM3(KK, JJ, II)=0.0
0051      LM9(KK, JJ, II)=0.0
0052      LM15(KK, JJ, II)=IC(KK, JJ, II)
0053      DO 50 NN=1, 3
0054      LM12(NN, KK, JJ, II)=0.0
0055      50 LM17(KK, JJ, II, NN)=ID(KK, JJ, II, NN)
0056      DO 51 KK=1, 3
0057      51 LM12(KK, KK, JJ, II)=PDMASS(JJ, II)
0058      LM20(JJ, II)=IH(JJ, II)
0059      LM3(JJ, JJ, II)=HCMASS(II)
0060      55 LM9(JJ, JJ, II)=HCMASS(II)
0061      60 CONTINUE
0062      RETURN
0063      END

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20.1 (AUG 71)

CS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NJKREF

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0002      SUREOLTIME FLX12
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *A ALPHA(3,3,3),ALPHA(3),PETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DELS(3,3,3),DEL6(3),DEL7(3),DELS(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *EIH(3),EIP(3),EH,EP,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGR(3,3),GXL(3,3),DD(500),
          *GRH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYR(3,3),GYRM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KBVH(3,3),KBVR(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM48(3,3),LM13B(3,3,3,3),KHG(6,18),
          *LM16R(3,3,3,3),KP(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MB(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PI THE T(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *RDMASS(3,3),PH(3),RI,RJ,RMASS,RRMASS,RTS(84),S11(3,3),S12(3),
          *S13(3,3,3),S14(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THE T(3),THE TD(3),PHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),LD(3,3,3),UDD(3,3,3),WCK1(3,3),WCK2(84,84),
          *WCK3(84),WCK4(3,3),WCK5(3,3),WCK6(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YEB(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
0010      DIMENSION W7(3,3)
          C
0011      KK=P
          C LM1
0012      CALL SMPY(PIGAM,MAST,LM1,3,3)
          C LM8,PARTIAL LM2
0013      CALL SCLA(LM2,0.,3,3)
0014      DO 70 II=1,3
0015      CALL GMADD(YC(1,II),UC(1,II),WCK1,3,1)
0016      CALL GAMMA(WCK4,WCK1)
0017      CALL SMPY(WCK4,-HCMASS(II),LM8(1,1,II),3,3)
0018      70 CALL GMADD(LM8(1,1,II),LM2,LM2,3,3)
0019      IF(NS.EQ.0.)GO TO 90
0021      CALL SMPY(M4,SMASS,WCK1,3,3)
0022      CALL GMADD(WCK1,LM2,LM2,3,3)
0023      90 IF(PASS.GT.C.)GO TO 90
0025      CALL GAMMA(GYB,YR)
0026      CALL SMPY(GYR,PMASS,GYRM,3,3)

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0027      90 CALL GMSUR(LM2,GYRM,LM2,3,3)
      C   LM4,LM5,COMPLETE LM2
0028      CALL SMPY(M1,PRMASS,LM4,3,3)
0029      DO 100 II=1, KK
0030      DO 100 JJ=1, 3
0031      CALL SMPY(M3(1,1,JJ,II),RDMASS(JJ,II),#OK1,3,3)
0032      CALL GMADD(LM4,WCK1,LM4,3,3)
0033      100 CALL SMPY(M2(1,1,II),RDMASS(JJ,II),LM5(1,1,JJ,II),3,3)
0034      CALL GMPRD(LM4,NTHET,WCK4,3,3,3)
0035      CALL GMPRD(WCK4,PIHET,WCK1,3,3,3)
0036      CALL GMADD(WCK1,LM2,LM2,3,3)
0037      CALL SMPY(LM4,1,LM4B,3,3)
0038      CALL GMPRD(LM4B,STHE,LM4,3,3,3)
      C   LM6
0039      CALL SMPY(M5,SMASS,LM6,3,3)
      C   LM7
0040      DO 110 II=1,3
0041      110 CALL SMPY(PIGAM,HCMASS(II),LM7(1,1,II),3,3)
      C   LM10,LM11,LM12,LM13
0042      CALL GMPRD(FITHE T,PIGAM,WCK1,3,3,3)
0043      CALL GMPRD(FITHE T,GYB,WCK4,3,3,3)
0044      DO 120 II=1, KK
0045      CALL GMPRD(AALPHA(1,1,II),WCK1,WCK5,3,3,3)
0046      CALL GMPRD(AALPHA(1,1,II),WCK4,WCK6,3,3,3)
0047      CALL GMPRD(AALPHA(1,1,II),FITHE T,W7,3,3,3)
0048      DO 120 JJ=1, 3
0049      CALL SMPY(WCK5,RDMASS(JJ,II),LM10(1,1,JJ,II),3,3)
0050      CALL SMPY(W7,RDMASS(JJ,II),WCK3,3,3)
0051      CALL GMPRD(WCK3,M3(1,1,JJ,II),LM13R(1,1,JJ,II),3,3,3)
0052      CALL SMPY(WCK6,-RDMASS(JJ,II),LM11(1,1,JJ,II),3,3)
0053      CALL GMPRD(LM13R(1,1,JJ,II),NTHET,WCK3,3,3,3)
0054      CALL GMPRD(WCK3,FITHE T,WCK2,3,3,3)
0055      CALL GMADD(WCK2,LM11(1,1,JJ,II),LM11(1,1,JJ,II),3,3)
0056      120 CALL GMPRD(LM13R(1,1,JJ,II),STHE,LM13(1,1,JJ,II),3,3,3)
      C   LM14
0057      DO 130 II=1,3
0058      130 CALL GMPRD(IC(1,1,II),PIPHIC(1,1,II),LM14(1,1,II),3,3,3)
      C   LM15
0059      DO 140 II=1, KK
0060      DO 140 JJ=1, 3
0061      CALL GMPRD(PIPHID(1,1,JJ,II),AALPHA(1,1,II),#OK1,3,3,3)
0062      CALL GMPRD(ID(1,1,JJ,II),WCK1,LM16R(1,1,JJ,II),3,3,3)
0063      140 CALL GMPRD(LM16R(1,1,JJ,II),STHE,LM15(1,1,JJ,II),3,3,3)
      C   LM18,LM19
0064      IF(NS.EQ.0.)GO TO 150
0065      CALL GMPRD(GPH,JT,WCK1,3,3,3)
0066      CALL GMPRD(WCK1,PIGAM,WCK4,3,3,3)
0067      CALL SMPY(WCK4,-SMASS,LM18,3,3)
0068      CALL GMPRD(WCK1,GYH,WCK4,3,3,3)
0069      CALL SMPY(WCK4,SMASS,LM19,3,3)
0070      CALL GMPRD(IH,JT,WCK1,3,3,3)
0071      CALL GMADD(LM19,WCK1,LM19,3,3)
      C   LM21,LM22,LM23
0072      150 CALL GMPRD(GXL,PIHET,WCK1,3,3,3)
0073      CALL GMPRD(WCK1,PIGAM,WCK4,3,3,3)
0074      CALL SMPY(WCK4,PRMASS,LM21,3,3)
0075      CALL GMPRD(WCK1,GYB,WCK4,3,3,3)
0076      CALL SMPY(WCK4,-PRMASS,LM22,3,3)

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0078 CALL GMPRD(IR,NTHET,WCK4,3,3,3)
0079 CALL GMPRD(WCK4,FITHET,WCK1,3,3,3)
0080 CALL GMADD(WCK1,LM22,LM22,3,3)
0081 CALL GMPRD(IR,STHE,LM23,3,3,3)
C LM24,LM25,LM26
0082 CALL SMPY(GYR,-HMASS,LM24,3,3)
0083 CALL SMPY(LM24,1.,WCK1,3,3)
0084 CALL SMPY(IA,1.,LM25,3,3)
0085 CALL SCLA(LM26,0.,3,3)
0086 IF(NS.EQ.C.)GO TO 160
0088 CALL GMSLB(YH,YB,WCK1,3,1)
0089 CALL GAMMA(WCK4,WCK1)
0090 CALL SMPY(WCK4,SMASS,WCK1,3,3)
0091 CALL GMADD(WCK1,LM24,LM24,3,3)
0092 CALL GMPRD(WCK1,M5,LM26,3,3,3)
0093 CALL GMPRD(WCK1,M4,WCK4,3,3,3)
0094 CALL GMADD(LM25,WCK4,LM25,3,3)
0095 CALL GMTRA(JT,WCK1,3,3)
0096 CALL GMPRD(WCK1,IH,WCK4,3,3,3)
0097 CALL GMADD(WCK4,LM26,LM26,3,3)
0098 CALL GMPRD(WCK4,JT,WCK5,3,3,3)
0099 CALL GMADD(LM25,WCK5,LM25,3,3)
0100 CALL GMPRD(WCK1,GRH,WCK4,3,3,3)
0101 CALL GMPRD(WCK4,JT,WCK1,3,3,3)
0102 CALL SMPY(WCK1,SMASS,WCK4,3,3)
0103 CALL GMSLB(LM24,WCK4,WCK1,3,3)
0104 CALL GMPRD(WCK4,GYH,WCK5,3,3,3)
0105 CALL GMADD(LM25,WCK5,LM25,3,3)
0106 160 CONTINUE
0107 CALL GMPRD(WCK1,FIGAN,LM24,3,3,3)
0108 RETURN
0109 END
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20.1 (AUG 71)

OS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINENCT=60,SIZE=0000,
SOURCE,RCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

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0002      SUBROUTINE FLX13
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELC(3),
          *EIH(3),FIR(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),G
          *GJH(3),GJP(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GRH(3,3),GWP(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYR(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(2,2),IR(3,3),IC(2,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KRVR(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4R(3,3),LM13R(3,3,3,3),KHG(6,18),
          *LM16R(3,3,3,3),KP(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(2,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MB(3),MBEAR(3),MC(3,3),ND(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),FIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PI THET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(2,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *RDMASS(3,3),RH(3),PI,FJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(2,3,3),SI4(2),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(2,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(2),THE TD(3),RHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WOK5(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),W2(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
          C SET UP THE RIGHT SIDE OF THE EGM
          C COMPUTE LOWER CASE PSI -STEP13
          C SI1
0010      CALL GAMMA(GWY,WYH)
0011      CALL GMPRD(GWY,GWY,GWYS,3,3,3)
0012      DD 10 II=1,3
0013      CALL GMADD(YC(1,II),UC(1,II),WOK1,3,1)
0014      CALL GMPRD(GWYS,WOK1,WOK4,3,3,1)
0015      CALL GMPRD(GWY,UCD(1,II),WCK1,3,3,1)
0016      CALL SMPY(WOK1,2,SI1(1,II),3,1)
0017      10 CALL GMADD(SI1(1,II),WCK4,SI1(1,II),3,1)
          C SI2
0018      CALL GAMMA(GWX,WXP)
0019      CALL GMPRD(GWX,GWX,WOK1,3,3,3)
0020      CALL GMPRD(WOK1,XL,WCK4,3,3,1)
0021      CALL GMTRA(PI THET,WOK1,3,3)
0022      CALL GMPRD(WOK1,WCK4,SI2,3,3,1)
0023      CALL GMPRD(GWYS,YR,WOK1,3,3,1)

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0024      CALL GMADD(SI2,WCK1,SI2,3,1)
C   SI3
0025      KK=F
0026      DO 20 II=1,KK
0027      CALL GAMMA(GWXRA(1,1,II),WXRA(1,II))
0028      CALL GMPRD(GWXPA(1,1,II),GWXRA(1,1,II),WCK4,3,3,3)
0029      DO 20 JJ=1,3
0030      CALL GMADD(UD(1,JJ,II),UD(1,JJ,II),WCK5,3,1)
0031      CALL GMPRD(WCK4,WCK5,WCK3,3,3,1)
0032      CALL GMPRD(GWXRA(1,1,II),UD(1,JJ,II),WCK6,3,3,1)
0033      CALL SMPY(WCK6,2,WCK5,3,1)
0034      CALL GMADD(WCK3,WCK5,WCK6,3,1)
0035      CALL GMPRD(W2(1,1,II),WCK6,WCK5,3,3,1)
0036      20 CALL GMADD(WCK1,WCK5,SI3(1,JJ,II),3,1)
C   SI4
0037      CALL SCIA(SI4,0,3,1)
0038      IF(NS.EQ.0.)GO TO 30
0040      CALL GMPRD(GWYS,YH,SI4,3,3,1)
0041      CALL GAMMA(GPSID,FSID)
0042      CALL GMPRD(GPSID,FBETA,WCK1,3,3,3)
0043      CALL GMPRD(WCK1,WYH,WCK4,3,3,1)
0044      CALL GMPRD(GRH,WCK4,WCK1,3,3,1)
0045      CALL GAMMA(GWR,WR)
0046      CALL GMPRD(GWR,GWF,WCK4,3,3,3)
0047      CALL GMPRD(WCK4,FR,WCK5,3,3,1)
0048      CALL GMADD(WCK5,WCK1,WCK4,3,1)
0049      CALL GMTRA(JT,WCK1,3,3)
0050      CALL GMPRD(WCK1,WCK4,WCK5,3,3,1)
0051      CALL GMSUB(SI4,WCK5,SI4,3,1)
0052      30 CONTINUE
0053      RETURN
0054      END

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20.1 (AUG 71)

05/360 FORTRAN H

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COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,RCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF
0002      SUBROUTINE FLX14
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CR(57,57),CH(18,18),CHG(6,18),DEL(3),DELC(3),
          *FTH(3),FIR(3),FH,FP,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),G
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GPH(3,3),GWR(3,3),GWX(3,3),GWXFA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYB(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KRVF(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM5(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4R(3,3),LM13R(3,3,3,3),KHG(6,18)
          *LM16R(3,3,3,3),KR(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MR(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PITHE T(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *RDMASS(3,3),RH(3),RI,RJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THE T(3),THE TD(3),FHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WOK6(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
          C CONTINUE TO SET UP RIGHT SIDE OF EDM
          C COMPUTE DELTA-STEP 14
          C DEL1,DEL2,DEL3
0010      KK=P
0011      CALL SCLA(DEL1,0.,3,1)
0012      CALL SCLA(WCK1,0.,3,3)
0013      DO 40 JJ=1,KK
0014      CALL GMPRD(AALPHA(1,1,JJ),PITHE T,WOK4,3,3,3)
0015      DO 40 II=1,3
0016      CALL SMPY(SI1(1,II,JJ),RDMASS(II,JJ),WOK5,3,1)
0017      CALL GMPRD(WCK4,WCK5,DEL3(1,II,JJ),3,3,1)
0018      40 CALL GMADD(DEL1,WCK5,DEL1,3,1)
0019      DO 41 JJ=1,3
0020      CALL SMPY(SI1(1,JJ),HCMASS(JJ),DEL2(1,JJ),3,1)
0021      41 CALL GMADD(WOK1,DEL2(1,JJ),WCK1,3,1)
0022      CALL GMADD(DEL1,WCK1,DEL1,3,1)
0023      CALL SMPY(SI2,RRMASS,WCK1,3,1)
0024      CALL GMADD(DEL1,WCK1,DEL1,3,1)

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0025     IF(NS.FQ.C.)GO TO 50
0027     CALL SMPY(SI4,SMASS,WCK1,3,1)
0028     CALL GMADD(DEL1,WCK1,DEL1,3,1)
0029     50 CONTINUE
C DEL4,DEL5
0030     DO 60 JJ=1,3
0031     CALL GMPRD(IC(1,1,JJ),WYC(1,JJ),WOK1,3,3,1)
0032     CALL GAMMA(WOK4,WYC(1,JJ))
0033     CALL GMPRD(WOK4,WCK1,DEL4(1,JJ),3,3,1)
0034     CALL GAMMA(WOK4,PHICD(1,JJ))
0035     CALL GMPRD(WOK4,WYH,WCK1,3,3,1)
0036     CALL GMPRD(IC(1,1,JJ),WCK1,WCK4,3,3,1)
0037     CALL GMSUB(DEL4(1,JJ),WCK4,DEL4(1,JJ),3,1)
0038     DO 60 II=1,KK
0039     CALL GAMMA(WOK1,WZ(1,JJ,II))
0040     CALL GMPRD(WOK1,ID(1,1,JJ,II),WOK4,3,3,3)
0041     CALL GMPRD(WOK4,WZ(1,JJ,II),DELS(1,JJ,II),3,3,1)
0042     CALL GAMMA(WCK1,PHIDD(1,JJ,II))
0043     CALL GMPRD(WCK1,AALPHA(1,1,II),WOK4,3,3,3)
0044     CALL GMPRD(WOK4,WXP,WCK1,3,3,1)
0045     CALL GMPRD(ID(1,1,JJ,II),WOK1,WOK4,3,3,1)
0046     60 CALL GMSUB(DELS(1,JJ,II),WCK4,DELS(1,JJ,II),3,1)
C DEL6
0047     IF(NS.EQ.C.)GO TO 70
0048     CALL GMPRD(IH,WR,WCK1,3,3,1)
0050     CALL GMPRD(GWP,WCK1,DEL6,3,3,1)
0051     CALL GMPRD(IH,GPSID,WCK1,3,3,3)
0052     CALL GMPRD(WCK1,PIBETA,WCK4,3,3,3)
0053     CALL GMPRD(WOK4,WYH,WCK1,3,3,1)
0054     CALL GMSUB(DEL6,WCK1,DEL6,3,1)
0055     CALL GMPRD(GRH,JT,WOK1,3,3,3)
0056     CALL GMPRD(WCK1,GWYS,WOK4,3,3,3)
0057     CALL GMPRD(WOK4,YH,WCK1,3,3,1)
0058     CALL SMPY(WCK1,SMASS,WOK4,3,1)
0059     CALL GMSUB(DEL6,WCK4,DEL6,3,1)
C DEL7
0060     70 CALL GMPRD(GWX,IF,WCK1,3,3,3)
0061     CALL GMPRD(WCK1,WXP,DEL7,3,3,1)
0062     CALL GMPRD(GXL,PITHT,WCK1,3,3,3)
0063     CALL GMPRD(WCK1,GWYS,WOK4,3,3,3)
0064     CALL GMPRD(WOK4,YB,WCK1,3,3,1)
0065     CALL SMPY(WCK1,PMASS,WCK4,3,1)
0066     CALL GMADD(DEL7,WCK4,DEL7,3,1)
C DEL8
0067     CALL GMPRD(GWY,IA,WCK1,3,3,3)
0068     CALL GMPRD(WCK1,WYH,DEL8,3,3,1)
0069     IF(NS.FQ.C.)GO TO 80
0071     CALL GMPRD(GWP,IH,WCK1,3,3,3)
0072     CALL GMPRD(WCK1,WF,WCK4,3,3,1)
0073     CALL GMPRD(IH,GPSID,WCK1,3,3,3)
0074     CALL GMPRD(WCK1,PIBETA,WCK5,3,3,3)
0075     CALL GMPRD(WCK5,WYH,WCK1,3,3,1)
0076     CALL GMSUB(WOK4,WCK1,WCK5,3,1)
0077     CALL GMTRA(JT,WOK1,3,3)
0078     CALL GMPRD(WCK1,WCK5,WCK4,3,3,1)
0079     CALL GMADD(DEL8,WCK4,DEL8,3,1)
0080     CALL GMSUB(YH,YB,WCK5,3,1)
0081     CALL GAMMA(WCK4,WCK5)

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```
0082 CALL GMPRD(WOK4,SI4,WCK5,3,3,1)
0083 CALL GMPRD(WCK1,GRH,WCK4,3,3,3)
0084 CALL GMPRD(WCK4,JT,WCK1,3,3,3)
0085 CALL GMPRD(WCK1,GWYS,WOK4,3,3,3)
0086 CALL GMPRD(WOK4,YH,WOK1,3,3,1)
0087 CALL GMSUR(WOK5,WCK1,WOK4,3,1)
0088 CALL SMPY(WCK4,SMAS,WCK1,3,1)
0089 CALL GMADD(DELB,WCK1,DELB,3,1)
0090 PD CONTINUE
0091 RETURN
0092 END
```

20.1 (AUG 71)

OS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,CFT=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

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0002     SUBROUTINE FLX15
0003     IMPLICIT REAL (A-Z)
0004     COMMON/FLX/
          *AALPHA(3,3,7),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DLTB,
          *QR(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *EIH(3),EIR(7),EH,EP,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),G
          *GJH(7),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GPH(3,3),GWR(3,3),GWX(3,3),GWXA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(7,3),GYR(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005     COMMON/FLX/
          *IA(7,3),IR(3,7),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KBVH(3,3),KBVP(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,7),LM3(3,3,7),LM4(3,7),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LMB(3,3),LM13B(3,3,3,3),KHG(6,18),
          *LM16R(3,3,3,3),KR(57,57),KH(18,18)
0006     COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MP(7),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIRETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(7,3),PI THET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(7),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007     COMMON/FLX/
          *RDMASS(3,7),RH(7),RI,RJ,RMASS,RFMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,7),SIG6(3),SIG7(7),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(3),THETD(3),FHC(3),STHE(3,3),SPIN
0008     COMMON/FLX/
          *UC(3,3),UCD(3,7),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WOK6(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
          *XL(7),XLL(84),XVEC(84),XVECD(84),XVECOD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009     DIMENSION MBS(3),MRV(3),KBV(3,3)
0010     INTEGER II,JJ,KK,LL,NN
          C COMPLETE THE RIGHT SID OF ECM
          C CALCULATE ELASTIC MOMENT AT DOCKING MECHANISM
          C MDOCK
0011         IF(NS.EQ.0.)GO TO 10
0012         CALL GMPRD(KHS,PSI,WOK1,3,3,1)
0013         CALL GMPRD(KHV,PSID,WCK4,3,3,1)
0014         CALL GMADD(WOK1,WCK4,WOK1,3,1)
0015         CALL SMPY(WCK1,-1.,MDOCK,3,1)
0016
0017     10 CONTINUE
          C CALCULATE BEARING TORQUES
          C MBS-SPRING MOMENT
0018         DLTB=ARCCS(CCS(THET(2))*CCS(THET(3)))
0019         MBS(1)=0.0
          C SPRING DISCONTINUITY
0020         IF(DLTB.GT.DELTB)GO TO 20
0021         KBS=-KBS1
0022
0023         GO TO 30

```



```

0024      20 KRS=(KRS2-KBS1)*DELTB/DLTE-KBS2
0025      30 MRS(2)=KRS*THET(2)
0026      MRS(3)=KRS*THET(3)
C        DAMPING MOMENT-MBV
C        DAMPING MATRIX -KBV
0027      CALL GMTRA(FITHE T,WCK1,3,3)
0028      CALL GMPRD(KRVR,WCK1,WCK4,3,3,3)
0029      CALL GMPRD(PI THE T,WCK4,WCK1,3,3,3)
0030      ETA1=1.-ETA
0031      CALL SMPY(WCK1,ETA1,WCK4,3,3)
0032      CALL SMPY(KBVH,ETA,WCK1,3,3)
0033      CALL GMADD(WCK1,WCK4,KBV,3,3)
0034      CALL GMPRD(PI THE T,WYH,WCK1,3,3,1)
0035      CALL GMSUB(WCK1,WXP,WCK1,3,1)
0036      CALL GMPRD(KRV,WCK1,MBV,3,3,1)
C        TOTAL BEARING MOMENT
0037      CALL GMADD(MRS,MBV,MBEAR,3,1)
C
C        CALCULATE ELASTIC FORCES AND MOMENTS DUE TO ROTOR ARM BENDING
C        SET UP TEMPORARY ROTOR STATE IN XC AND DERIV. IN XCD
0038      DO 40 II=1,3
0039      XC(II)=DEL(II)
0040      40 XCD(II)=DELD(II)
0041      LL=P
0042      DO 50 KK=1,LL
0043      DO 50 JJ=1,3
0044      DO 50 II=1,3
0045      NN=3+6*(JJ-1)+18*(KK-1)
0046      XC(II+NN)=UD(II,JJ,KK)
0047      XC(II+NN+3)=PHID(II,JJ,KK)
0048      XCD(II+NN)=LDD(II,JJ,KK)
0049      50 XCD(II+NN+3)=PHIDD(II,JJ,KK)
0050      NN=39
0051      IF(P,FO,3.) NN=57
0052      DO 55 II=1,NN
0053      WCK3(II)=0.0
0054      DO 55 KK=1,NN
0055      55 WCK3(II)=WCK3(II)+KR(II,KK)*XC(KK)+CR(II,KK)*XCD(KK)
0056      60 DO 62 II=1,3
0057      62 MP(II)=WCK3(II)
0058      DO 65 KK=1,LL
0059      DO 65 JJ=1,3
0060      DO 65 II=1,3
0061      NN=3+6*(JJ-1)+18*(KK-1)
0062      FD(II,JJ,KK)=WCK3(II+NN)
0063      65 MD(II,JJ,KK)=WCK3(II+NN+3)
C        CALCULATE FORCES AND MOMENTS DUE TO HUB BENDING
C        SET UP TEMP HUB STATE IN XC AND ITS DERIV IN XCD
0064      DO 70 JJ=1,3
0065      DO 70 II=1,3
0066      NN=6*(JJ-1)
0067      XC(II+NN)=UC(II,JJ)
0068      XCD(II+NN)=UCD(II,JJ)
0069      XC(II+NN+3)=PHIC(II,JJ)
0070      70 XCD(II+NN+3)=PHICD(II,JJ)
0071      CALL GMPRD(KH,XC,WCK2,18,18,1)
0072      CALL GMPRD(CH,XCD,WCK3,18,18,1)
0073      CALL GMADD(WCK3,WCK2,WCK3,18,1)
0074

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0075      DO 75 JJ=1,3
0076      DO 75 II=1,3
0077      NN=6*(JJ-1)
0078      FC(II,JJ)=WCK3(II+NN)
0079      75 MC(II,JJ)=WCK3(II+NN+3)
0080      CALL GMPRD(KHG,XC,WCK2,6,18,1)
0081      CALL GMPRD(CHG,XCD,WCK3,6,19,1)
0082      CALL GMADD(WCK3,WCK2,WCK3,6,1)
0083      DO 80 II=1,3
0084      FG(II)=WCK3(II)
0085      80 MG(II)=WCK3(II+3)
C
C CALCULATE THE SIGMA'S
C SIG1
0086      CALL GMPRD(LM4B,RHC,WCK1,3,3,1)
0087      CALL SMPY(DEL1,-1.,SIG1,3,1)
0088      CALL GMSUB(SIG1,WCK1,SIG1,3,1)
C SIG2
0089      KK=P
0090      DO 90 JJ=1,3
0091      CALL SMPY(DEL2(1,JJ),-1.,WCK1,3,1)
0092      CALL GMSUB(WCK1,FC(1,JJ),SIG2(1,JJ),3,1)
C SIG4
0093      CALL GMSUB(TAUC(1,JJ),DEL4(1,JJ),WCK1,3,1)
0094      CALL GMPRD(PIPHIC(1,1,JJ),MC(1,JJ),WCK4,3,3,1)
0095      CALL GMSUB(WCK1,WCK4,SIG4(1,JJ),3,1)
C SIG3
0096      DO 90 II=1,3
0097      IF(JJ.GT.KK)GO TO 90
0099      CALL SMPY(DEL3(1,II,JJ),-1.,WCK1,3,1)
0100      CALL GMSUB(WCK1,FD(1,II,JJ),WCK1,3,1)
0101      CALL GMPRD(LM13B(1,1,II,JJ),RHC,WCK4,3,3,1)
0102      CALL GMSUB(WCK1,WCK4,SIG3(1,II,JJ),3,1)
C SIG5
0103      CALL GMSUB(TAUD(1,II,JJ),DEL5(1,II,JJ),WCK1,3,1)
0104      CALL GMPRD(PIPHID(1,1,II,JJ),MD(1,II,JJ),WCK4,3,3,1)
0105      CALL GMSUB(WCK1,WCK4,WCK1,3,1)
0106      CALL GMPRD(LM16B(1,1,II,JJ),RHC,WCK4,3,3,1)
0107      CALL GMSUB(WCK1,WCK4,SIG5(1,II,JJ),3,1)
0108      90 CONTINUE
C SIG6
0109      IF(NS.EQ.0.)GO TO 95
0111      CALL GMADD(TAUS,MDOCK,SIG6,3,1)
0112      CALL GMSUB(SIG6,DEL6,SIG6,3,1)
0113      GO TO 100
0114      95 IF(PASS.GT.C.)GO TO 100
0116      DO 97 II=1,3
0117      97 SIG6(II)=C.0
C SIG7
0118      100 CALL GMADD(TALL,MBEAP,SIG7,3,1)
0119      CALL GMSUB(SIG7,MR,SIG7,3,1)
0120      CALL GMSUB(SIG7,DFL7,SIG7,3,1)
0121      CALL GMPRD(IR,RHC,WCK1,3,3,1)
0122      CALL GMSUB(SIG7,WCK1,SIG7,3,1)
C SIG8
0123      IF(PASS.GT.C.)GO TO 200
0125      CALL GMSUB(YG,YR,WCK1,3,1)
0126      CALL GAMMA(GYGR,WCK1)

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0127      200 CALL GMPRD(GYGR,FG,WOK1,3,3,1)
0128      CALL GMSUB(TAUA,WOK1,SIG8,3,1)
0129      CALL GMSUR(SIG8,DFL8,SIG8,3,1)
0130      CALL GMSUR(SIG8,NG,SIG8,3,1)
0131      CALL GMTRA(FITHE1,WOK1,3,3)
0132      CALL GMPRD(WOK1,MPEAR,WCK4,3,3,1)
0133      CALL GMSUR(SIG8,WCK4,SIG8,3,1)
0134      IF(NS.EQ.0.)GO TO 210
0135      CALL GMTRA(JT,WOK1,3,3)
0136      CALL GMPRD(WOK1,TAUS,WCK4,3,3,1)
0137      CALL GMADD(SIG8,WCK4,SIG8,3,1)
0138
0139      210 CONTINUE
0140      RETURN
0141      END
```

20.1 (AUG 71)

DS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,PCD,NOLIST,DECK,LOAD,MAP,NOFDIT, ID,NOXREF

```

0002      SUBROUTINE PREINT
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
      *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
      *DEL4(3,3),DELS(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
      *CF(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
      *FH(3),EIR(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH
      *GJH(3),GJR(3),GPSID(3,3),GR,GYGR(3,3),GXL(3,3),DD(500),
      *GRH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
      *GWYS(3,3),GYR(3,3),GYRM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
      *IA(3,3),IR(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
      *KPVH(3,3),KRV(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
      *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
      *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
      *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
      *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
      *LM24(3,3),LM25(3,3),LM26(3,3),LM4R(3,3),LM13R(3,3,3,3),KHG(6,18),
      *LM16R(3,3,3,3),KF(57,57),KH(18,18)
0006      COMMON/FLX/
      *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
      *MR(3),MREAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
      *N,NS,P,PASS,PIRFTA(3,3),PIGAN(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
      *PIPSI(3,3),PIPHET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
      *PHIDD(3,3,3),PSI(3),PSIC(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
      *PDMASS(3,3),PH(3),PI,RJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
      *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
      *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
      *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
      *THET(3),THEID(3),FHC(3),STHE(3,3),SPIN
0008      COMMON/FLX/
      *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
      *WOK3(84),WOK4(3,3),WOK5(3,3),WOK6(3,3),WXR(3),WXRA(3,3),
      *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3),
      *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD0(84),YC(3,3),YB(3),YE3(3),
      *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,NN
0010      INTEGER IJ,N,M,MM
0011      INTEGER NROT,NHUB
0012      DIMENSION LM27(3,3,3,3)
      C FORM THE COMPLETE MASS MATRIX LAM(84,84)
      C
      C MODIFY MASS TERMS FOR DELETIONS
      C      NS=0. SHUTTLE NOT ATTACHED,NS=1. SHUTTLE ATTACHED
0013      IF(NS.EQ.1.)GO TO 10
0015      DO 5 II=1,3
0016      DO 4 JJ=1,3
0017      LM18(II,JJ)=0.0
0018      LM19(II,JJ)=0.0
0019      4 LM20(II,JJ)=0.
0020      5 LM20(II,II)=250000.
0021      10 CONTINUE
      C      MODIFY FOR A TWO ARMED FCTOR
0022      IF(P.EQ.3.)GO TO 20
0024      DO 12 II=1,3

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0025      DO 12 JJ=1,3
0026      DO 12 KK=1,3
0027      LM10(II,JJ,KK,3)=0.
0028      LM11(II,JJ,KK,3)=0.
0029      LM12(II,JJ,KK,3)=0.
0030      LM13(II,JJ,KK,3)=0.
0031      LM16(II,JJ,KK,3)=0.
0032      LM27(II,JJ,KK,3)=0.0
0033      12 LM17(II,JJ,KK,3)=0.0
0034      20 CONTINUE
C SET UP MASS MATRIX
0035      DO 25 II=1,84
0036      DO 25 JJ=1,84
0037      25 LAM(II,JJ)=0.0
0038      M=84.
C CALCULATE LM27
0039      KK=P
0040      CALL GMPRD(NTHET,PITHET,WCK1,3,3,3)
0041      DO 35 JJ=1,KK
0042      DO 35 II=1,3
0043      35 CALL GMPRD(LM16R(1,1,II,JJ),WCK1,LM27(1,1,II,JJ),3,3,3)
0044      CALL STACK(LM1,LAM,3,3,84,1,1)
0045      CALL STACK(LM2,LAM,3,3,84,1,4)
0046      CALL STACK(LM6,LAM,3,3,84,1,7)
0047      CALL STACK(LM7,LAM,3,9,84,1,10)
0048      CALL STACK(LM4,LAM,3,3,84,1,28)
0049      CALL STACK(LM5,LAM,3,9,84,1,31)
0050      CALL STACK(LM5(1,1,1,2),LAM,3,9,84,1,42)
0051      CALL STACK(LM5(1,1,1,3),LAM,3,9,84,1,67)
0052      CALL STACK(LM24,LAM,3,3,84,4,1)
0053      CALL STACK(LM25,LAM,3,3,84,4,4)
0054      CALL STACK(LM26,LAM,3,3,84,4,7)
0055      CALL STACK(LM18,LAM,3,3,84,7,1)
0056      CALL STACK(LM19,LAM,3,3,84,7,4)
0057      CALL STACK(LM20,LAM,3,3,84,7,7)
0058      DO 50 JJ=1,3
0059      DO 50 KK=1,3
0060      DO 50 II=1,3
0061      NN=II+3*(KK-1)+9
0062      LAM(NN,JJ)=LM7(II,JJ,KK)
0063      LAM(NN,JJ+3)=LM8(II,JJ,KK)
0064      LAM(NN+9,JJ+3)=LM14(II,JJ,KK)
0065      MM=3*(KK-1)+9+JJ
0066      LAM(NN,MM)=LM9(II,JJ,KK)
0067      50 LAM(NN+9,MM+9)=LM15(II,JJ,KK)
0068      CALL STACK(LM21,LAM,3,7,84,28,1)
0069      CALL STACK(LM22,LAM,3,3,84,28,4)
0070      CALL STACK(LM23,LAM,3,3,84,28,28)
0071      DO 60 IJ=1,3
0072      DO 60 JJ=1,3
0073      DO 60 KK=1,3
0074      DO 60 II=1,3
0075      NN=II+3*(KK-1)+18*(IJ-1)+30
0076      LAM(NN,JJ)=LM10(II,JJ,KK,IJ)
0077      LAM(NN,JJ+3)=LM11(II,JJ,KK,IJ)
0078      LAM(NN,JJ+27)=LM13(II,JJ,KK,IJ)
0079      LAM(NN+9,JJ+27)=LM16(II,JJ,KK,IJ)
0080      LAM(NN+9,JJ+3)=LM27(II,JJ,KK,IJ)

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0081      MM=JJ+3*(KK-1)+18*(IJ-1)+30
0082      LAM(NN,MM)=LM12(II,JJ,KK,IJ)
0083      60 LAM(NN+9,MM+9)=LM17(II,JJ,KK,IJ)
0084      CALL GENM
C FORM REDUCED MASS MATRIX - TRANS(MODE SHAPES).LAM.(MODE SHAPES)
C FIRST CALCULATE TRANS(MODE SHAPES).LAM, STORE IN WOK2
C NOTE THE MATRIX MULT. IS CARRIED OUT BELOW TO
C AVOID UNNECESSARY ZERO MULTIPLICATIONS

0085      N=9.+NPOT+NHUR
0086      DO 70 II=1,N
0087      DO 70 JJ=1,84
0088      70 WOK2(II,JJ)=0.0
C ROW 1 THRU 8
0089      DO 75 II=1,3
0090      DO 71 JJ=1,18
0091      71 WOK2(II,JJ)=LAM(II,JJ)
0092      DO 72 JJ=28,39,1
0093      72 WOK2(II,JJ)=LAM(II,JJ)
0094      DO 73 JJ=49,57,1
0095      73 WOK2(II,JJ)=LAM(II,JJ)
0096      DO 75 JJ=67,75,1
0097      75 WOK2(II,JJ)=LAM(II,JJ)
0098      DO 76 II=4,8,1
0099      DO 76 JJ=1,9
0100      76 WOK2(II,JJ)=LAM(II,JJ)
0101      DO 77 II=4,6,1
0102      DO 77 JJ=10,84,1
0103      77 WOK2(II,JJ)=LAM(II,JJ)
C ROWS 9 THRU 8+NO. OF HUB MODES=8+NHUR
0104      IF(NHUR.EQ.0)GO TO 100
0106      DO 85 II=1,NHUR
0107      DO 80 JJ=1,3
0108      DO 80 KK=1,9
0109      80 WOK2(II+8,JJ)=WOK2(II+8,JJ)+PHIMH(KK,II)*LAM(KK+9,JJ)
0110      DO 85 JJ=4,6,1
0111      DO 85 KK=1,18
0112      85 WOK2(II+8,JJ)=WOK2(II+8,JJ)+PHIMH(KK,II)*LAM(KK+9,JJ)
0113      NN=0
0114      DO 95 LL=1,6
0115      DO 90 II=1,NHUR
0116      DO 90 JJ=1,3
0117      DO 90 KK=1,3
0118      90 WOK2(II+8,JJ+9+NN)=WOK2(II+8,JJ+9+NN)+PHIMH(KK+NN,II)*LAM(KK+9+NN
      *JJ+9+NN)
0119      95 NN=NN+3
C ROWS 9+NHUR THRU N
0120      100 IF(NPOT.EQ.0)GO TO 140
0122      NN=NHUR+8
0123      LL=57
0124      IF(P.EQ.2.)LL=39
0126      DO 115 II=1,NPOT
0127      DO 110 JJ=1,6
0128      DO 105 KK=1,LL
0129      105 WOK2(II+NN,JJ)=WOK2(II+NN,JJ)+PHIMH(KK,II)*LAM(KK+27,JJ)
0130      110 CONTINUE
0131      DO 111 JJ=7,27,1
0132      DO 111 KK=1,3
0133      111 WOK2(II+NN,JJ)=WOK2(II+NN,JJ)+PHIMH(KK,II)*LAM(KK+27,JJ)

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0134      DO 112 JJ=31,84,1
0135      DO 112 KK=1,3
0136      112 WOK2(II+NN, JJ)=WOK2(II+NN, JJ)+PHIMR(KK, II)*LAM(KK+27, JJ)
0137      DO 115 JJ=28,30,1
0138      DO 115 KK=1,LL
0139      115 WOK2(II+NN, JJ)=WOK2(II+NN, JJ)+PHIMR(KK, II)*LAM(KK+27, JJ)
0140      LL=0
0141      DO 135 II=1,19
0142      DO 130 II=1, NROT
0143      DO 130 JJ=1,3
0144      DO 130 KK=1,3
0145      130 WOK2(II+NN, JJ+LL+30)=WOK2(II+NN, JJ+LL+30)+PHIMR(KK+LL+3, II)*LAM(
          *+30+LL, JJ+LL+30)
0146      135 LL=LL+7
0147      140 CONTINUE
C
C      MULTIPLY MODE SHAPES BY WOK2 TO COMPLETE THE REDUCED
C      MASS MATRIX-MASS(N,N)
0148      DO 205 II=1,N
0149      DO 205 JJ=1,N
0150      205 MASS(II, JJ)=0.0
C      COL. 1 THRU 8
0151      DO 207 JJ=1,8
0152      DO 207 II=1,N
0153      207 MASS(II, JJ)=WOK2(II, JJ)
C      COL. 9 THRU NHUB PLUS 8
0154      NN=NHUB+8
0155      IF(NHUB.EQ.0)GO TO 215
0157      DO 210 II=1,N
0158      DO 210 JJ=1, NHUB
0159      DO 210 KK=1,18
0160      210 MASS(II, JJ+8)=MASS(II, JJ+8)+WOK2(II, KK+9)*PHIMR(KK, JJ)
0161      215 IF(NPOT.EQ.0)GO TO 225
0163      LL=30
0164      IF(P.FO.3.)LL=57
C      COL.(NHUB+8) THRU N
0166      DO 220 II=1,6
0167      DO 220 JJ=1, NROT
0168      DO 216 KK=1,LL
0169      216 MASS(II, JJ+NN)=MASS(II, JJ+NN)+WOK2(II, KK+27)*PHIMR(KK, JJ)
0170      220 CONTINUE
0171      DO 222 II=1, NROT
0172      DO 222 JJ=1, NROT
0173      DO 222 KK=1,LL
0174      222 MASS(II+NN, JJ+NN)=MASS(II+NN, JJ+NN)+WOK2(II+NN, KK+27)*PHIMR(KK, JJ)
0175      225 CONTINUE
C
C      INVERT MASS AND STORE IN MASS
0176      CALL SPEVN(MASS, N, WOK3, M, NIX)
C      SET UP RIGHT SIDE FORCING FUNCTION
0177      DO 200 II=1,3
0178      RTS(II)=SIG1(II)
0179      RTS(II+3)=SIG2(II)
0180      RTS(II+6)=SIG6(II)*NS
0181      DO 190 JJ=1,3
0182      NN=JJ+9+3*(II-1)
0183      RTS(NN)=SIG2(JJ, II)
0184      190 RTS(NN+9)=SIG4(JJ, II)

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0185     RTS(II+27)=SIG7(II)
0186     DO 195 JJ=1,3
0187     NN=30+JJ+3*(II-1)
0188     RTS(NN)=SIG3(JJ,II,1)
0189     RTS(NN+9)=SIG5(JJ,II,1)
0190     RTS(NN+18)=SIG3(JJ,II,2)
0191     RTS(NN+27)=SIG5(JJ,II,2)
0192     IF(P.EQ.2.)GO TO 192
0194     RTS(NN+36)=SIG3(JJ,II,3)
0195     RTS(NN+45)=SIG5(JJ,II,3)
0196     GO TO 195
0197     192 RTS(NN+36)=C.0
0198     RTS(NN+45)=C.0
0199     195 CONTINUE
0200     200 CONTINUE
0201     CALL GFNR
      C   FORM REDUCED RIGHT SIDE AND STORE IN WOK3
0202     DO 300 II=1,N
0203     3000 WOK3(II)=C.0
0204     DO 300 II=1,8
0205     300 WOK3(II)=RTS(II)
0206     IF(NHUR.EQ.0)GO TO 320
0208     DO 310 II=1,NHUR
0209     DO 310 KK=1,18
0210     310 WOK3(II+8)=WOK3(II+8)+FHI*H(KK,II)*RTS(KK+9)
0211     320 NN=NHUR+8
0212     IF(NPOT.EQ.0)GO TO 350
0214     LL=39
0215     IF(P.EQ.3.)LL=57
0217     DO 330 II=1,NPOT
0218     DO 330 KK=1,LL
0219     330 WOK3(II+NN)=WOK3(II+NN)+FHI*H(KK,II)*RTS(KK+27)
0220     350 CONTINUE
      C   CALCULATE THE SECOND DERIVATIVES
0221     DO 4000 II=1,N
0222     4000 XVECDD(II)=C.0
0223     DO 400 II=1,N
0224     DO 400 KK=1,N
0225     400 XVECDD(II)=XVECDD(II)+MASS(II,KK)*WOK3(KK)
0226     RETURN
0227     END

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20.1 (AUG 71)

05/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,CET=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT,ID,NOXREF

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0002      SUBROUTINE INTIG
0003      IMPLICIT REAL (A-Z)
0004      COMMON/TIME/T,DT,NSYM,ANTTWC
0005      COMMON/FLX/
          *A ALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *EIH(3),EIR(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GI
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GRH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYR(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0006      COMMON/FLX/
          *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KRVR(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),<HG(6,18),
          *LM16B(3,3,3,3),KF(57,57),KH(18,18)
0007      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MB(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PI THET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSIC(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0008      COMMON/FLX/
          *RDMASS(3,3),RH(3),PI,PJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(3),THE TD(3),PHC(3),STHE(3,3),SPIN
0009      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),#OK1(3,3),#OK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WCK6(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3)
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0010      DIMENSION NP(84)
0011      DIMENSION NFASS(4),XS(4),XSL(4),XCDI(84),XCI(84)
0012      INTEGER NPASS,NRCT,NHUB,NSYM
0013      INTEGER II,JJ,KK,LL,NN,NF
0014      INTEGER IJ,A,M
          C PERFORM INTEGRATIONS
0015      IF(PASS.GT.C.)GO TO 10
0017      DO 5 II=1,N
0018      5 NP(II)=0.0
0019      DO 6 II=1,4
0020      6 NPASS(II)=0.0
0021      DO 10C II=1,84
          C INITIAL PHYSICAL COORDINATES
0022      XC(II)=DD(300+II)
0023      10C XCD(II)=DD(400+II)
0024      XC(28)=DEL(1)
0025      XC(29)=DEL(2)
0026      XC(30)=DEL(3)

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0027      XCD(28)=DELD(1)
0028      XCD(29)=DELD(2)
0029      XCD(30)=DELD(3)
0030      IF(ANYTWO.EQ.0.)GO TO 108
0032      DO 109 II=1,57
0033      109 XCD(II+27)=PHIMP(II,NSYM+2)*DD(430)/PHIMR(3,NSYM+2)
0034      108 CONTINUE
      C      INITIAL CONDITIONS OF MODAL COORDINATES
      C      COMPUTE THE INVERSE OF (PHI(TRANS)PHI) AND STORE IN WOK2
0035      DO 110 JJ=1,N
0036      DO 110 II=1,N
0037      110 WOK2(II,JJ)=0.0
0038      DO 115 II=1,8
0039      115 WOK2(II,II)=1.0
0040      IF(NHUR.EQ.0)GO TO 125
0042      DO 120 II=1,NHUR
0043      DO 120 JJ=1,NHUR
0044      DO 120 KK=1,18
0045      120 WOK2(II+8,JJ+8)=WOK2(II+8,JJ+8)+PHIMH(KK,II)*PHIMH(KK,JJ)
0046      125 IF(NROT.EQ.0)GO TO 135
0048      NN=NHUR+8
0049      LL=39
0050      IF(P.EQ.3.)LL=57
0052      DO 130 II=1,NROT
0053      DO 130 JJ=1,NROT
0054      DO 130 KK=1,LL
0055      130 WOK2(II+NN,JJ+NN)=WOK2(II+NN,JJ+NN)+PHIMR(KK,II)*PHIMR(KK,JJ)
0056      135 CONTINUE
      C      INVERT
0057      CALL SREVN1(WOK2,N,WOK3,M,NIX)
      C      COMPUTE (PHI(TRANS).XC) AND STORE IN WOK3
0058      DO 1400 II=1,N
0059      1400 WOK3(II)=0.0
0060      DO 140 II=1,8
0061      140 WOK3(II)=XC(II)
0062      IF(NHUR.EQ.0)GO TO 150
0064      DO 145 II=1,NHUR
0065      DO 145 KK=1,18
0066      145 WOK3(II+8)=WOK3(II+8)+PHIMH(KK,II)*XC(KK+9)
0067      150 IF(NROT.EQ.0)GO TO 160
0069      DO 155 II=1,NROT
0070      DO 155 KK=1,LL
0071      155 WOK3(II+NN)=WOK3(II+NN)+PHIMR(KK,II)*XC(KK+27)
0072      160 CONTINUE
      C      CALCULATE XVEC(0)=WCK2.WCK3
0073      DO 1700 II=1,N
0074      1700 XVEC(II)=0.0
0075      DO 170 II=1,N
0076      DO 170 KK=1,N
0077      170 XVEC(II)=XVEC(II)+WCK2(II,KK)*WCK3(KK)
      C
      C      COMPUTE (PHI(T).XCD) AND STORE IN WOK3
0078      DO 2400 II=1,N
0079      2400 WOK3(II)=0.0
0080      DO 240 II=1,8
0081      240 WOK3(II)=XCD(II)
0082      IF(NHUR.EQ.0)GO TO 250
0084      DO 245 II=1,NHUR

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0085      DO 245 KK=1,18
0086      245 WOK3(I1+8)=WOK3(I1+8)+PHIMH(KK,I1)*XCD(KK+9)
0087      250 IF(NRPT.EQ.0)GO TO 260
0089      DO 255 I1=1,NRPT
0090      DO 255 KK=1,LL
0091      255 WOK3(I1+NN)=WOK3(I1+NN)+PHIMR(KK,I1)*XCD(KK+27)
0092      260 CONTINUE
C      CALCULATE XVECD(0)=WCK2, WOK3
0093      DO 2700 I1=1,N
0094      2700 XVECD(I1)=0.0
0095      DO 270 I1=1,N
0096      DO 270 KK=1,N
0097      270 XVECD(I1)=XVECD(I1)+WCK2(I1,KK)*WOK3(KK)
C
0098      10 DO 20 I1=1,N
0099      20 CALL PIEMAN(XVEC(I1),XVECD(I1),XVECD(I1),DT,NP(I1),XCDI(I1),XCI(I1)
      *I),XLL(I1))
C
C      TRANSFORM MODAL COORDINATES INTO PHYSICAL COORDINATES
C      FIRST DERIVATIVES
C      CALCULATE (PHI, XVECD) AND STORE IN WOK3
0100      DO 3000 I1=1,84
0101      3000 WOK3(I1)=0.0
0102      DO 300 I1=1,8
0103      300 WOK3(I1)=XVECD(I1)
0104      WOK3(9)=0.
0105      IF(NHUR.EQ.0)GO TO 320
0107      DO 310 I1=1,18
0108      DO 310 KK=1,NHUR
0109      310 WOK3(I1+9)=WOK3(I1+9)+PHIMH(I1,KK)*XVECD(KK+8)
0110      GO TO 330
0111      320 DO 325 I1=1,18
0112      325 WOK3(I1+9)=0.0
0113      330 IF(NRPT.EQ.0)GO TO 340
0115      NN=NHUR+8
0116      DO 335 I1=1,LL
0117      DO 335 KK=1,NRPT
0118      335 WOK3(I1+27)=WOK3(I1+27)+PHIMR(I1,KK)*XVECD(KK+NN)
0119      GO TO 345
0120      340 DO 343 I1=27,84,1
0121      343 WOK3(I1)=0.0
0122      345 DO 50 I1=1,3
0123      ZAD(I1)=WOK3(I1)
0124      WYH(I1)=WOK3(I1+3)
0125      PSID(I1)=WOK3(I1+6)
0126      DO 40 JJ=1,3
0127      NN=9+JJ+3*(I1-1)
0128      UCD(JJ,I1)=WOK3(NN)
0129      40 PHICD(JJ,I1)=WOK3(NN+9)
0130      DELD(I1)=WOK3(I1+27)
0131      DO 45 JJ=1,3
0132      NN=27+JJ+3*(I1-1)
0133      UDD(JJ,I1,1)=WOK3(NN)
0134      PHIDD(JJ,I1,1)=WOK3(NN+9)
0135      UDD(JJ,I1,2)=WOK3(NN+18)
0136      PHIDD(JJ,I1,2)=WOK3(NN+27)
0137      IF(P.EQ.2)GO TO 45
0139      UDD(JJ,I1,3)=WOK3(NN+36)

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0140 PHIDD (JJ,II,3)=WCK3(NN+45)
0141 45 CONTINUE
0142 50 CONTINUE
C
C STATE VECTOR
C CALCULATE (PHI,XVEC) AND STORE IN WOK3
0143 DO 350 II=1,84
0144 350 WOK3(II)=0.0
0145 DO 350 II=1,84
0146 350 WOK3(II)=XVEC(II)
0147 WOK3(9)=0.
0148 IF(NHUB.EQ.0)GO TO 370
0150 DO 360 II=1,18
0151 DO 360 KK=1,NHUB
0152 360 WOK3(II+9)=WOK3(II+9)+PHIMH(II,KK)*XVEC(KK+8)
0153 GO TO 380
0154 370 DO 375 II=1,18
0155 375 WOK3(II+9)=0.0
0156 380 IF(NROT.EQ.0)GO TO 390
0158 NN=NHUB+8
0159 DO 385 II=1,LL
0160 DO 385 KK=1,NROT
0161 385 WOK3(II+27)=WOK3(II+27)+PHIMF(II,KK)*XVEC(KK+NN)
0162 GO TO 395
0163 390 DO 393 II=27,84,1
0164 393 WOK3(II)=0.0
0165 395 DO 70 II=1,3
0166 ZA(II)=WOK3(II)
0167 PSI(II)=WOK3(II+6)
0168 DEL(II)=WOK3(II+27)
0169 DO 60 JJ=1,3
0170 NN=9+JJ+3*(II-1)
0171 UC(JJ,II)=WCK3(NN)
0172 PHIC(JJ,II)=WCK3(NN+9)
0173 UD(JJ,II,1)=WCK3(NN+21)
0174 PHID(JJ,II,1)=WCK3(NN+30)
0175 UD(JJ,II,2)=WCK3(NN+39)
0176 PHID(JJ,II,2)=WCK3(NN+48)
0177 IF(P.EQ.2.)GO TO 60
0179 UD(JJ,II,3)=WCK3(NN+57)
0180 PHID(JJ,II,3)=WCK3(NN+66)
0181 60 CONTINUE
0182 70 CONTINUE
C INTEGRATE THETA(1)
0183 CALL STLJES(THET(1),THETA(1),DT,NPASS(1),XS(1),XSL(1))
C INTEGRATE GAMDA
0184 DO 112 II=1,3
0185 112 CALL STLJES(GAMDA(II),GAMDA(II),DT,NPASS(II+1),XS(II+1),XSL(II+1))
C FORM THETA(2),THETA(3)
0186 CALL GMPRD(PITHET,WYH,WCK1,3,3,1)
0187 THETA(1)=DELD(1)+SPIN-WCK1(1,1)
0188 THETA(2)=DELD(2)
0189 THETA(3)=DELD(3)
0190 THET(2)=DEL(2)
0191 THET(3)=DEL(3)
C FORM GAMDA
0192 CALL GMPRD(SGAM,WYH,GAMDA,3,3,1)
0193 RETURN

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20.1 (AUG 71)

CS/360 FORTRAN H

COMPILER OPTIONS = NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,BCD,NOLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

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0002      C(SRPLTIME GENM
0003      IMPLICIT REAL (A-Z)
0004      COMMON/FLX/
          *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
          *DFL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELTB,
          *CP(57,57),CH(18,18),CHG(6,18),DEL(3),DELD(3),
          *FIR(3),EIR(3),FH,FR,FTA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3),GH,
          *GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
          *GPH(3,3),GWP(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
          *GWYS(3,3),GYB(3,3),GYPN(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0005      COMMON/FLX/
          *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS2,
          *KRVH(3,3),KRVP(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
          *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
          *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
          *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3,3),
          *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
          *LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),KHG(6,18),
          *LM16B(3,3,3,3),KR(57,57),KH(18,18)
0006      COMMON/FLX/
          *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
          *MR(3),MBEAP(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
          *N,NS,P,PASS,PIBETA(3,3),FIGAM(3,3),PIPHIC(3,3,3),PIPHID(3,3,3,3),
          *PIPSI(3,3),PIPHET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
          *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0007      COMMON/FLX/
          *PDMASS(3,3),PH(3),RI,RJ,PMASS,FRMASS,RTS(84),SI1(3,3),SI2(3),
          *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
          *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
          *SUMJ(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
          *THET(3),THETD(3),PHO(3),STHE(3,3),SPIN
0008      COMMON/FLX/
          *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
          *WOK3(84),WOK4(3,3),WOK5(3,3),WCK5(3,3),WR(3),WXR(3),WXRA(3,3),
          *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XC(3,3,3),XN(3,3),
          *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3),
          *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0009      INTEGER II,JJ,KK,LL,MM
0010      DIMENSION A(3,3),B(3,3),D(3,3,3),C(3,3),E(3,3,3),F(3,3,3,3),
          *G(3,3,3,3),H(3,3,3),GX(3,84),GY(3,57)
0011      CALL GMTRA(FIGAM,A,3,3)
0012      S2=SIN(GAM(2))
0013      S3=SIN(GAM(3))
0014      C2=COS(GAM(2))
0015      C3=COS(GAM(3))
0016      C(1,1)=C3*C2
0017      C(2,1)=S3
0018      C(3,1)=0.
0019      C(1,2)=-S3*C2
0020      C(2,2)=C3
0021      C(3,2)=0.
0022      C(1,3)=S2
0023      C(2,3)=0.
0024      C(3,3)=1.
0025      CALL GMPRD(C,GYP,B,3,3,3)
          C FW U000      C.0000      0.0000
    
```

```

0026      DO 1 II=1,3
0027      CALL GMADD(YC(1,II),UC(1,II),WOK1,3,1)
0028      CALL GMSUR(WOK1,YB,WOK4,3,1)
0029      CALL GAMMA(WOK1,WOK4)
0030      1 CALL GMPRD(C,WOK1,D(1,1,II),3,3,3)
      C FORM E
0031      KK=P
0032      DO 5 II=1,KK
0033      CALL GMTRA(PI THET,WOK1,3,3)
0034      CALL GMTRA(AALPHA(1,1,II),WOK4,3,3)
0035      CALL GMPRD(WOK1,WOK4,WOK5,3,3,3)
0036      CALL GMPRD(C,WOK5,E(1,1,II),3,3,3)
      C FORM H
0037      CALL GMTRA(STHE,WOK6,3,3)
0038      CALL GMPRD(WOK6,WOK4,H(1,1,II),3,3,3)
      C FORM F
0039      CALL GMPRD(AALPHA(1,1,II),PITHET,WOK4,3,3,3)
0040      CALL GMPRD(WOK4,YB,WOK6,3,3,1)
0041      CALL GMPRD(B,WOK5,WOK1,3,3,3)
0042      DO 5 JJ=1,3
0043      CALL GMADD(XD(1,JJ,II),UD(1,JJ,II),WOK4,3,1)
0044      CALL GMADD(WOK6,WOK4,WOK5,3,1)
0045      CALL GAMMA(WOK2,WOK5)
0046      CALL GMPRD(E(1,1,II),WOK2,WOK5,3,3,3)
0047      CALL GMSUB(WOK5,WOK1,F(1,1,JJ,II),3,3)
      C FORM G
0048      CALL GAMMA(WOK5,WOK4)
0049      5 CALL GMPRD(H(1,1,II),WOK5,G(1,1,JJ,II),3,3,3)
      C CREATE THE NECESSARY PORTION OF THE TRANSFORMATION MATRIX
0050      CALL GMTRA(PI THET,WOK1,3,3)
0051      CALL GMPRD(C,WOK1,WOK4,3,3,3)
0052      DO 50 JJ=1,3
0053      DO 50 II=1,3
0054      GX(II,JJ)=R(II,JJ)
0055      GX(II,JJ+3)=C(II,JJ)
0056      GX(II,JJ+6)=0.
0057      GX(II,JJ+27)=WOK4(II,JJ)
0058      DO 10 LL=1,3
0059      NN=C+2*(JJ-1)
0060      GX(LL,II+NN)=D(LL,II,JJ)
0061      GX(LL,II+NN+9)=C(LL,II)
0062      GX(LL,II+NN+21)=F(LL,II,JJ,1)
0063      GX(LL,II+NN+30)=F(LL,II,1)
0064      GX(LL,II+NN+39)=F(LL,II,JJ,2)
0065      GX(LL,II+NN+48)=F(LL,II,2)
0066      IF(KK.EQ.2) GO TO 15
0068      GX(LL,II+NN+57)=F(LL,II,JJ,3)
0069      GX(LL,II+NN+66)=F(LL,II,3)
0070      15 GY(LL,II+NN-6)=G(LL,II,JJ,1)
0071      GY(LL,II+NN+3)=H(LL,II,1)
0072      GY(LL,II+NN+12)=G(LL,II,JJ,2)
0073      GY(LL,II+NN+21)=H(LL,II,2)
0074      IF(KK.EQ.2) GO TO 10
0076      GY(LL,II+NN+30)=G(LL,II,JJ,3)
0077      GY(LL,II+NN+39)=H(LL,II,3)
0078      10 CONTINUE
0079      50 GY(II,JJ)=STHE(JJ,II)
      C

```

```

C   TRANSFORM MASS MATRIX
0080      LL=84
0081      NN=57
0082      IF(KK.EQ.3)GO TO 80
0084      LL=66
0085      NN=70
0086      80 DO 100 JJ=1,LL
0087          DO 100 II=1,LL
0088      100 MASS(II,JJ)=LAM(II,JJ)
0089          DO 150 JJ=1,LL
0090          DO 150 II=1,3
0091          LAM(II,JJ)=C.
0092          DO 150 KK=1,7
0093      150 LAM(II,JJ)=LAM(II,JJ)+A(II,KK)*MASS(KK,JJ)
0094          DO 200 JJ=1,LL
0095          DO 200 II=1,3
0096          LAM(II+3,JJ)=0.
0097          DO 200 KK=1,LL
0098      200 LAM(II+3,JJ)=LAM(II+3,JJ)+GX(II,KK)*MASS(KK,JJ)
0099          DO 300 JJ=1,LL
0100          DO 300 II=1,3
0101          LAM(II+27,JJ)=0.0
0102          DO 300 KK=1,NN
0103      300 LAM(II+27,JJ)=LAM(II+27,JJ)+GY(II,KK)*MASS(KK+27,JJ)
0104      RETURN
0105      ENTRY GENR

C   TRANSFORM RIGHT SIDE
0106      DO 500 II=1,LL
0107      500 XC(II)=RTS(II)
0108          DO 600 JJ=1,3
0109          RTS(JJ)=0.0
0110          DO 600 KK=1,3
0111      600 RTS(JJ)=RTS(JJ)+A(JJ,KK)*XC(KK)
0112          DO 700 JJ=1,3
0113          RTS(JJ+3)=0.0
0114          DO 700 KK=1,LL
0115      700 RTS(JJ+3)=RTS(JJ+3)+GX(JJ,KK)*XC(KK)
0116          DO 800 JJ=1,3
0117          RTS(JJ+27)=0.0
0118          DO 800 KK=1,NN
0119      800 RTS(JJ+27)=RTS(JJ+27)+GY(JJ,KK)*XC(KK+27)
0120      RETURN
0121      END

```

20.1 (AUG 71)

OS/360 FORTRAN H

COMPILER OPTIONS - NAME= MAIN,OPT=02,LINECNT=60,SIZE=0000K,
SOURCE,PCD,NCLIST,DECK,LOAD,MAP,NOEDIT, ID,NOXREF

```

0002      SUBROUTINE CMG
0003      IMPLICIT REAL (A-Z)
0004      COMMON/TIME/T,DT,NSYM,ANTWO
0005      COMMON/FLX/
      *AALPHA(3,3,3),ALPHA(3),BETA(3),DEL1(3),DEL2(3,3),DEL3(3,3,3),
      *DEL4(3,3),DEL5(3,3,3),DEL6(3),DEL7(3),DEL8(3),DELT8,
      *CR(57,57),CH(18,18),CHG(6,18),DEL(3),DELC(3),
      *EIH(3),EIR(3),EH,ER,ETA,FC(3,3),FD(3,3,3),FG(3),GAM(3),GAMD(3)
      *GJH(3),GJR(3),GPSID(3,3),GR,GYGB(3,3),GXL(3,3),DD(500),
      *GRH(3,3),GWR(3,3),GWX(3,3),GWXRA(3,3,3),GWY(3,3),GWYC(3,3,3),
      *GWYS(3,3),GYB(3,3),GYBM(3,3),GYH(3,3),HCMASS(3),HI,HJ,HMASS
0006      COMMON/FLX/
      *IA(3,3),IB(3,3),IC(3,3,3),ID(3,3,3,3),IH(3,3),JT(3,3),KBS1,KBS
      *KBVH(3,3),KBVR(3,3),KHS(3,3),KHV(3,3),LAM(84,84),LM1(3,3),
      *LM2(3,3),LM3(3,3,3),LM4(3,3),LM5(3,3,3,3),LM6(3,3),LM7(3,3,3),
      *LM8(3,3,3),LM9(3,3,3),LM10(3,3,3,3),LM11(3,3,3,3),LM12(3,3,3,3),
      *LM13(3,3,3,3),LM14(3,3,3),LM15(3,3,3),LM16(3,3,3,3),LM17(3,3,3),
      *LM18(3,3),LM19(3,3),LM20(3,3),LM21(3,3),LM22(3,3),LM23(3,3),
      *LM24(3,3),LM25(3,3),LM26(3,3),LM4B(3,3),LM13B(3,3,3,3),KHG(6,18)
      *LM16B(3,3,3,3),KR(57,57),KH(18,18)
0007      COMMON/FLX/
      *M,M1(3,3),M2(3,3,3),M3(3,3,3,3),M4(3,3),M5(3,3),MAST,MASS(84,84),
      *MB(3),MBEAR(3),MC(3,3),MD(3,3,3),MDOCK(3),MG(3),NROT,NHUB,
      *N,NS,P,PASS,PIBETA(3,3),PIGAM(3,3),PIPHI((3,3,3),PIPHID(3,3,3),
      *PIPSI(3,3),PIPHET(3,3),PHIC(3,3),PHICD(3,3),PHID(3,3,3),
      *PHIDD(3,3,3),PSI(3),PSID(3),PHIMR(57,57),PHIMH(18,18),NTHET(3,3)
0008      COMMON/FLX/
      *RDMASS(3,3),RH(3),RI,RJ,RMASS,RRMASS,RTS(84),SI1(3,3),SI2(3),
      *SI3(3,3,3),SI4(3),SIG1(3),SIG2(3,3),SIG3(3,3,3),SIG4(3,3),
      *SIG5(3,3,3),SIG6(3),SIG7(3),SIG8(3),SGAM(3,3),SMASS,STHET(3,3),
      *SUM1(3,3,3),TAUA(3),TAUC(3,3),TAUD(3,3,3),TAJL(3),TAUS(3),
      *THET(3),THETD(3),RHC(3),STHE(3,3),SPIN
0009      COMMON/FLX/
      *UC(3,3),UCD(3,3),UD(3,3,3),UDD(3,3,3),WOK1(3,3),WOK2(84,84),
      *WOK3(84),WOK4(3,3),WOK5(3,3),WOK6(3,3),WR(3),WXR(3),WXRA(3,3),
      *WXY(3),WYC(3,3),WYH(3),WZ(3,3,3),XC(84),XCD(84),XD(3,3,3),XN(3,3,3),
      *XL(3),XLL(84),XVEC(84),XVECD(84),XVECD(84),YC(3,3),YB(3),YE3(3,3),
      *YG(3),YH(3),ZA(3),ZAD(3),ZV3(3,3)
0010      COMMON/CONT/
      *TC(3),TAUF,START,STCP,H,WNG,ZETG,BDR,B,FE,BDE,BDEF,BDEDD,BDED,
      *BIAS,WAIT
0011      DIMENSION XCOR(3),XLAS(3),NF(3)
0012      INTEGER II,NP
      C THIS SUBROUTINE DEVELOPS THE CONTROL TORQUES,TC, FOR WOBBLE DAMPING
      C THE WOBBLE DAMPING TECHNIQUE CONSISTS FOR A SGCMG THAT IS DRIVEN
      C THAT IT'S ANGULAR MOMENTUM VECTOR LAGS THE WOBBLE VECTOR BY 90 DEGREES
      C ANGULAR RATES OF BODY THAT CMG IS WITHIN
0013      WP1=WXP(1)
0014      WP2=WXP(2)
0015      WP3=WXP(3)
0016      IF(PASS.GT.C)GO TO 400
0018      DONE=1.
0019      AL=0.
0020      BDED=-WP1
0021      BDEF=C.
0022      B=0.0

```



```

0023      DO 1 II=1,3
0024      1 NR(II)=0.0
      C TEST FOR START AND STOP
0025      400 IF(DONE.EQ.C.)GO TO 431
0027      IF(SQRT(WR2**2+WR3**2)-START)430,430,432
0028      432 DONE=C.
0029      GO TO 41
0030      431 IF(SQRT(WR2**2+WR3**2)-STOP)410,410,41
0031      410 DONE=1.
0032      430 BDP=-WR1
0033      XCOR(1)=BDP
0034      XLAS(1)=0.
0035      RDEFD=0.
0036      IF(R.GE.0.)GO TO 415
0038      R=6.2831853+R
0039      XCOR(2)=6.2831853+XCOR(2)
0040      415 CONTINUE
0041      GO TO 40
0042      41 IF(ABS(WR2)-.0000001)42,42,15
0043      15 XY=ABS(WR3)/ABS(WR2)
0044      FE=ATAN(XY)
0045      GO TO 43
0046      42 FE=1.5707963
0047      43 IF(WR2)18,17,17
0048      17 IF(WR3)50,50,48
0049      48 RDE=FE
      C TRANSFER FROM QUADA TO QUAD1
0050      IF(AL.EQ.1.)GO TO 480
0052      AL=0.
0053      GO TO 36
0054      480 RDEF=RDEF-6.2831853
0055      XCOR(7)=XCOR(3)-6.2831853
0056      AL=0.
0057      GO TO 36
0058      50 RDE=6.2831853-FE
      C TRANSFER FROM QUADU TO QUAD4
0059      IF(AL.EQ.0.)GO TO 490
0061      AL=1.
0062      GO TO 36
0063      490 RDEF=RDEF+6.2831853
0064      XCOR(3)=XCOR(3)+6.2831853
0065      AL=1.
0066      GO TO 36
0067      18 IF(WR3)32,31,31
0068      31 RDE=3.1415926-FE
0069      AL=-1.
0070      GO TO 36
0071      32 RDE=3.1415926+FE
0072      AL=-1.
      C LOW PASS FILTER
0073      36 IF(TAUF)601,601,605
0074      601 RDEF=RDE
0075      BDEFF=RDEF
0076      BDEFD=0.0
0077      GO TO 610
0078      605 RDEFD=(RDE-BDEF)/TAUF
      C BIAS THE COMMANDED ANGLE (BY BIA)FC COMPENSATE FOR FILTER LAG.
      C BIAS=0. FOR NO BIAS. BIAS=-CR+WCBLE FEQU. MINUS FOR MINI

```

```

0079      BJA=BIAS*TAUF
0080      RDEB=RDEFF+BJA

C
CONSTRAN FILTER OUTPUT BTWN 0 AND 2PI
0081      IF(RDEB.GT.6.2831853)GO TO 495
0083      IF(RDEB.LT.0.)GO TO 496
0085      RDEFF=RDEB
0086      GO TO 610
0087      495 RDEFF=RDEB-6.2831853
0088      GO TO 610
0089      496 RDEFF=6.2831853+RDEB
C DYNAMICS-TORQUE MOTOR+CYMBAL
0090      610 IF(B)530,532,532
0091      530 R=6.2831853+B
0092      XCOR(2)=XCOR(2)+6.2831853
0093      GO TO 535
0094      532 IF(R.LE.6.2831853)GO TO 535
0096      R=B-6.2831853
0097      XCOR(2)=XCOR(2)-6.2831853
0098      535 ER=RDEFF-R
0099      IF(ABS(ER).LE.3.1415926)GO TO 305
0101      ER1=6.2831853-ABS(ER)
0102      ER=-SIGN(ER1,ER)

C WAIT OPTION FOR ACTIVE CONTROL
0103      305 IF(WAIT.EQ.0.)GO TO 30
0105      IF(ABS(ER).GT.WAIT)GO TO 430
0107      30 BDEDD=ER*WNG**2-(2.*ZETG*WNG)*BDED
0108      32 IF(ABS(BDED)-1.) 34,34,33
0109      33 BDR=SIGN(1.,BDED)
0110      GO TO 40
0111      34 BDR=BDED
0112      40 CONTINUE

C LIMIT NEGATIVE RATE TO MINUS SPIN
0113      620 IF(BDED.LT.-WR1)BDR=-WR1
C 90 DEG H-LAG-LAW      R=ANGLE BTWN INNER GMBL AXIS AND THE 2 AXIS
0115      420 TC(2)=(-WR1-BDR)*H*COS(B)
0116      TC(3)=(-WR1-BDR)*H*SIN(B)
0117      TC(1)=(WR1*SIN(B)+WR2*CCS(B))*H
0118      CALL STLJFS(BDED,BDEDD,DT,NP(1),XCOR(1),XLAS(1))
0119      CALL STLJFS(B,BDR,DT,NP(2),XCOR(2),XLAS(2))
0120      CALL STLJES(BDEF,BDEFD,DT,NP(3),XCOR(3),XLAS(3))
C APPLY TORQUES TO BODY THAT CNG IS LOCATED WITHIN
0121      TAUL(1)=TC(1)
0122      TAUL(2)=TC(2)
0123      TAUL(3)=TC(3)
0124      RETURN
0125      END

```

SUBROUTINE RIEMAN(Y,Y1,Y2,DT,NPASS,Y1C,YC,Y2L)

THIS SUBROUTINE IS A MODIFIED-ADAMS INTEGRATION FOR A SECOND ORDER DIFFERENTIAL EQUATION. THE VARIABLES USED ARE AS FOLLOWS --

- * Y- DEPENDENT VARIABLE AS PREDICTED BY THIS INTEGRATION
- * Y1- FIRST DERIVATIVE OF Y (ALSO PREDICTED HERE)
- * Y2- SECOND DERIVATIVE OF Y FROM MAIN PROGRAM (PER DIF. EQ.)
- DT- INDEPENDENT VARIABLE
- * NPASS- 0 INITIALLY AND CHANGED TO 1 BY THIS SUBROUTINE
- * Y1C- CORRECTED VALUE OF Y1 COMPUTED FROM Y2L AND Y2
- * YC- CORRECTED VALUE OF Y COMPUTED FROM Y1L AND Y1
- * Y1L- LAST VALUE OF Y1
- * Y2L- LAST VALUE OF Y2

*NOTE- ALL THE VARIABLES WITH AN ASTERISK MUST BE SAVED IN THE CALLING PROGRAM. THE CALLING PROGRAM MUST INITIALIZE NPASS, Y,Y1 AND DT. Y2 IS COMPUTED IN THE CALLING PROGRAM FROM THE GOVERNING DIFFERENTIAL EQUATION. A SAMPLE CALLING SEQUENCE IS.

```

DO 1 I=1,10
X2DOT(I)=-2.*ZETA(I)*OMEGAN(I)*XIDOT(I)-OMEGAN(I)**2*X(I)+U(I)
CALL RIEMAN (X(I),XIDOT(I),X2DOT(I),.001,NP(I),XCOR(I),XC(I),XIL(I),
1 ),X2L(I) )
SOLVING THE TEN DIFFERENTIAL EQUATIONS...

```

$$\frac{D^2X}{DT^2} - I + 2Z \frac{DX}{DT} - I + O X = U \quad \text{FOR } I=1,10$$

```

IF(NPASS)5,3,5
3 Y2L=Y2
Y1C=Y1
YC=Y
NPASS=1
GO TO 6
5 YC=YC+Y1C*DT+(Y2+2.*Y2L)/6.*DT**2
Y1C=Y1C+DT*(Y2+Y2L)/2.
6 Y=YC+Y1C*DT+DT**2*(4.*Y2-Y2L)/6.
Y1=Y1C+DT*(3.*Y2-Y2L)/2.
Y2L=Y2
RETURN

```

FIGURE 10.14 STLJES

```

SUBROUTINE STLJES(Y,Y1,DT,NPASS,YC,Y1L)
IF ( NPASS .NE. 0 ) GO TO 5
Y1L=Y1
YC=Y
NPASS=1
GO TO 6
5 YC=YC+DT*(Y1+Y1L)/2.
6 Y=YC+DT*(3.*Y1-Y1L)/2.
Y1L=Y1
RETURN
END

```

REI0001
REI0002
REI0003
REI0004
REI0005
REI0006
REI0007
REI0008
REI0009
REI0010
REI0011
REI0012
REI0013
REI0014
REI0015
REI0016
REI0017
REI0018
REI0019
REI0020
REI0021
REI0022
REI0023
REI0024
REI0025
REI0026
REI0027
REI0028
REI0029
REI0030
REI0031
REI0032
REI0033
REI0034
REI0035
REI0036
REI0037
REI0038
REI0039
REI0040
REI0041
REI0042
REI0043
REI0044
STL0001
STL0002
STL0003
STL0004
STL0005
STL0006
STL0007
STL0008
STL0009
STL0010
STL0011


```

C SUBROUTINE ABCP1
SUBROUTINE ABCP1(A,PI,ANG,K)
THIS ROUTINE GENERATES THE 3X3 MATRICIES A AND PI
C
C A          3X3 REAL MATRIX A
C B          3X3 REAL MATRIX B
C C          3X3 REAL MATRIX C
C PI         3X3 REAL MATRIX PI
C K - CONTROL LOGIC
C   - 0 CALCULATE A MATRIX
C   - 1 CALCULATE A,PI MATRIX
C
C ANG          3 ELEMENT ARRAY OF ANGLES
C
C DIMENSION TEMP(9),A(9),B(9),C(9),ANG(3),PI(9)
C DO 1 I=2,4
1 A(I)=0.
C1=DCOS(DBLE(ANG(1)))
S1=DSIN(DBLE(ANG(1)))
A(7)=0.
A(1)=1.
A(5)=C1
A(6)=-S1
A(8)=S1
A(9)=C1
IF(K.EQ.0) GO TO 100
C DO 2 I=2,8,2
2 B(I)=0.
C2=DCOS(DBLE(ANG(2)))
S2=DSIN(DBLE(ANG(2)))
E(1)=C2
E(3)=S2
B(5)=1.
B(7)=-S2
E(9)=C2
DO 3 I=6,8
3 C(I)=0.
C3=DCOS(DBLE(ANG(3)))
S3=DSIN(DBLE(ANG(3)))
C(3)=0.
C(1)=C3
C(2)=-S3
C(4)=S3
C(5)=C3
C(9)=1.0
CALL GMPRC (C,B,TEMP,3,3,3)
CALL GMPRC (TEMP,A,PI,3,3,3)
100 RETURN
END

```

FLX0001
FLX0010
FLX0015
FLX0016
FLX0017
FLX0018
FLX0019
FLX0020
FLX0014
FLX0053
FLX0056
FLX0057
FLX0058
FLX0059
FLX0060
FLX0061
FLX0064
FLX0067
FLX0068
FLX0069
FLX0070
FLX0071
FLX0074
FLX0077
FLX0078
FLX0079
FLX0080
FLX0081
FLX0082
FLX0085
FLX0051

```

C SUBROUTINE SMPY
C PURPOSE
C   MULTIPLY EACH ELEMENT OF A MATRIX BY A SCALAR TO FORM A
C   RESULTANT MATRIX
C
C USAGE
C   CALL SMPY(A,C,R,N,M)
C
C DESCRIPTION OF PARAMETERS
C   A - NAME OF INPUT MATRIX
C   C - SCALAR
C   R - NAME OF OUTPUT MATRIX
C   N - NUMBER OF ROWS IN MATRIX A AND R
C   M - NUMBER OF COLUMNS IN MATRIX A AND R
C
SUBROUTINE SMPY(A,C,R,N,M)
DIMENSION A(1),R(1)
IT=N*M
DO 1 I=1, IT
1 R(I)=A(I)*C
RETURN
END

```

FIGURE 10.19 GMADD

```

C SUBROUTINE GMADD
C PURPOSE
C   ADD TWO GENERAL MATRICES TO FORM RESULTANT GENERAL MATRIX
C
C USAGE
C   CALL GMADD(A,B,R,N,M)
C
C DESCRIPTION OF PARAMETERS
C   A - NAME OF FIRST INPUT MATRIX
C   B - NAME OF SECOND INPUT MATRIX
C   R - NAME OF OUTPUT MATRIX
C   N - NUMBER OF ROWS IN A,B,R
C   M - NUMBER OF COLUMNS IN A,B,R
C
SUBROUTINE GMADD(A,B,R,N,M)
DIMENSION A(1),B(1),R(1)
NM=N*M
DO 10 I=1,NM
10 R(I)=A(I)+B(I)
RETURN
END

```

C SUBROUTINE SCLA

C PURPOSE

C SET EACH ELEMENT OF A MATRIX EQUAL TO A GIVEN SCALAR

C USAGE

C CALL SCLA(A,C,N,M)

C DESCRIPTION OF PARAMETERS

C A - NAME OF INPUT MATRIX

C C - SCALAR

C N - NUMBER OF ROWS IN MATRIX A

C M - NUMBER OF COLUMNS IN MATRIX A

C SUBROUTINE SCLA(A,C,N,M)

C DIMENSION A(1)

C IT=N*M

C DO 1 I=1, IT

C 1 A(I)=C

C RETURN

C END

FIGURE 10.21 GMTRA

C SUBROUTINE GMTRA

C PURPOSE

C TRANSPOSE A GENERAL MATRIX

C USAGE

C CALL GMTRA(A,R,N,M)

C DESCRIPTION OF PARAMETERS

C A - NAME OF MATRIX TO BE TRANSPOSED

C R - NAME OF RESULTANT MATRIX

C N - NUMBER OF ROWS OF A AND COLUMNS OF R

C M - NUMBER OF COLUMNS OF A AND ROWS OF R

C SUBROUTINE GMTRA(A,R,N,M)

C DIMENSION A(1),R(1)

C IR=0

C DO 10 I=1,N

C IJ=I-N

C DO 10 J=1,M

C IJ=IJ+N

C IR=IR+1

C 10 R(IR)=A(IJ)

C RETURN

C END

```

C   SUBROUTINE GMPRD
C   PURPOSE
C       MULTIPLY TWO GENERAL MATRICES TO FORM A RESULTANT GENERAL
C       MATRIX
C
C   USAGE
C       CALL GMPRD(A,B,R,N,M,L)
C
C   DESCRIPTION OF PARAMETERS
C       A - NAME OF FIRST INPUT MATRIX
C       B - NAME OF SECOND INPUT MATRIX
C       R - NAME OF OUTPUT MATRIX
C       N - NUMBER OF ROWS IN A
C       M - NUMBER OF COLUMNS IN A AND ROWS IN B
C       L - NUMBER OF COLUMNS IN B

```

```

SUBROUTINE GMPRD(A,B,R,N,M,L)
DIMENSION A(1),B(1),R(1)
IR=0
IK=-M
DO 10 K=1,L
IK=IK+M
DO 10 J=1,N
IR=IR+1
JI=J-N
IB=IK
R(IR)=0
DO 10 I=1,M
JI=JI+N
IB=IB+1
10 R(IR)=R(IR)+A(JI)*B(IB)
RETURN
END

```

FIGURE 10.23 GMSUB

```

C   SUBROUTINE GMSUB
C   PURPOSE
C       SUBTRACT ONE GENERAL MATRIX FROM ANOTHER TO FORM RESULTANT
C       MATRIX
C
C   USAGE
C       CALL GMSUB(A,B,R,N,M)
C
C   DESCRIPTION OF PARAMETERS
C       A - NAME OF FIRST INPUT MATRIX
C       B - NAME OF SECOND INPUT MATRIX
C       R - NAME OF OUTPUT MATRIX
C       N - NUMBER OF ROWS IN A,B,R
C       M - NUMBER OF COLUMNS IN A,B,R

```

```

SUBROUTINE GMSUB(A,B,R,N,M)
DIMENSION A(1),B(1),R(1)
NM=N*M
DO 10 I=1,NM
10 R(I)=A(I)-B(I)
RETURN
END

```


SUBROUTINE SREVN1(A,M,LOC,MID,NIX)

PLX0087

```

C
C   MATRIX INVERSION
C
C   A-CONVENTIONAL FORTRAN DOUBLE ARRAY CONTAINING MATRIX TO BE INVERTED
C   M- MATRIX ORDER
C   LOC- SINGLE ARRAY DIMENSIONED AT LEAST TO M
C   MID- FIRST DIMENSION OF A, NOT LESS THAN M
C   NIX- ERROR INDICATOR, SET TO ZERO AFTER SUCCESSFUL EXECUTION.
C
      DIMENSION A(MID,1)
      INTEGER LOC(1)
100  N = M
      DO 190 K = 1,N
      PIVOT = 0.0
      DO 120 I = K,N
      IF (PIVOT - ABS(A(I,K))) 110,110,120
110  PIVOT = ABS(A(I,K))
      L = I
120  CONTINUE
      IF (PIVOT) 140,130,140
130  NIX = -1
      GO TO 210
140  LOC(K) = L
      DO 150 J = 1,N
      TEMP1 = A(K,J)
      A(K,J) = A(L,J)
150  A(L,J) = TEMP1
      TEMP1 = A(K,K)
      A(K,K) = 1.0
      DO 160 J = 1,N
160  A(K,J) = A(K,J)/TEMP1
      DO 190 I = 1,N
      IF (I - K) 170,190,170
170  TEMP1 = -A(I,K)
      A(I,K) = 0.0
      DO 180 J = 1,N
180  A(I,J) = A(I,J) + TEMP1*A(K,J)
190  CONTINUE
      DO 200 K = 1,N
      NK = N - K
      L = LOC(NK+1)
      DO 200 I = 1,N
      TEMP1 = A(I,NK+1)
      A(I,NK+1) = A(I,L)
200  A(I,L) = TEMP1
      NIX = 0
210  RETURN
      ENC

```

```
SUBROUTINE WILLY(NAME,P,NR,NC,SR,ER,SC,EC)
```

```
C THIS SUBROUTINE WILL PRINT OUT A MATRIX STORED IN AS NR BY NC FORTRAN ARRAY
```

```
C THE ELEMENTS PRINTED OUT ARE BETWEEN ROWS SR AND ER AND COLUMNS SC AND EC
```

```
C A MAXIMUM OF TEN COLUMNS WILL BE PRINTED TO A LINE
```

```
C THE MATRIX WILL BE SEGMENTED INTO SUB-MATRICES OF TEN COLUMNS EACH
```

```
C THE ARGUMENTS OF THIS SUBROUTINE ARE
```

```
NAME - A DOUBLEWORD CONTAINING THE NAME TO BE PRINTED ABOVE THE MATRIX
```

```
- IN LITERAL FORM I.E. 'MATRIX A'
```

```
P - THE FORTRAN ARRAY
```

```
NR - THE FIRST DIMENSION OF P , AN INTEGER
```

```
NC - THE SECOND DIMENSION OF P , AN INTEGER
```

```
SR - THE FIRST ROW OF P TO BE PRINTED , AN INTEGER
```

```
ER - THE LAST ROW OF P TO BE PRINTED , AN INTEGER
```

```
SC - THE FIRST COLUMN OF P TO BE PRINTED , AN INTEGER
```

```
EC - THE LAST COLUMN OF P TO BE PRINTED , AN INTEGER
```

```
REAL NAME(2),P(NR,NC)
```

```
INTEGER SR,ER,SC,EC
```

```
NB=SC-10
```

```
NL=SC-1
```

```
10 NL=MINO(NL+10,EC)
```

```
NB=NB+10
```

```
WRITE(6,20)NAME,(I,I=NB,NL)
```

```
20 FORMAT('0',7X,2A4//' ROW COLUMN',I3,9I12)
```

```
DO 40 I=SR,ER
```

```
WRITE(6,30)I,(P(I,J),J=NB,NL)
```

```
30 FORMAT(I4,5X,10G12.5)
```

```
40 CONTINUE
```

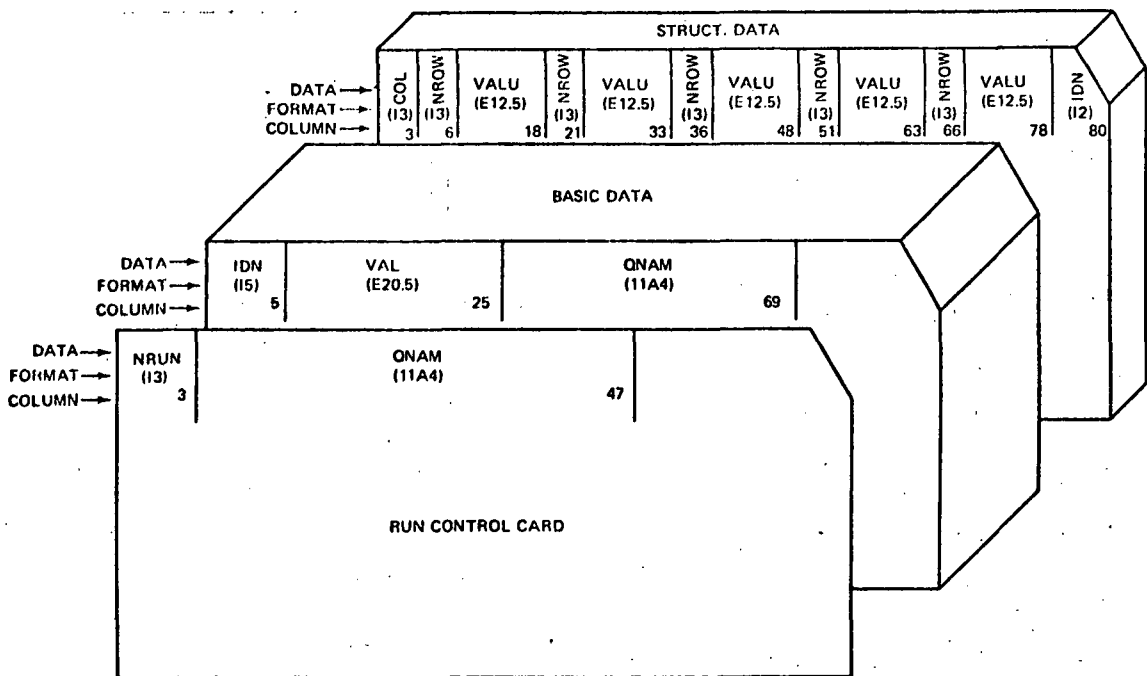
```
IF(NL .LT. EC) GO TO 10
```

```
RETURN
```

```
END
```

10.1.3 Operating Instructions

Program operation is governed by three groups of input data cards: RUN CONTROL CARD, BASIC DATA CARDS, and STRUCTURAL DATA CARDS. The required order for these groups is shown below along with the required format and corresponding information provided by each data card.



RUN CONTROL CARD - The first data card for each set of data representing a single run. NRUN is an integer variable defining the number of the run to be executed. If NRUN should equal zero all running is terminated. Multiple runs are accomplished by stacking data sets. The last data set must contain a RUN CONTROL CARD only, with NRUN = 0; this will terminate the day's running. NRUN also serves another logical function and that is to control the input of STRUCTURAL DATA. Specifically, if NRUN = 1 a complete set of STRUCTURAL DATA must be read in; if NRUN > 1 then the STRUCTURAL DATA used for the previous run is acceptable.

QNAM is an alphanumeric variable used to identify the run to be executed.

BASIC DATA - Defines: vehicle geometry, mass properties, initial state, CMG control logic and hardware characteristics, and general simulation timing logic for printing, plotting and integration.

Each data card defines a single input variable whose value is VAL, and is identified by it's identification number IDN. All BASIC DATA are initialized to zero internal to the program, so that only values other than zero need be read-in. Subsequent runs require data cards only to define changes to BASIC DATA. A blank card is required at the end of the BASIC DATA group so as to terminate data read-in. Subsequent runs that do not require changes to BASIC DATA still require a blank card in the BASIC DATA position of the data set.

Table 10.2 provides a correlation between identification numbers and the corresponding Basic Data variables. These variables have been defined in Table 10.1, however, a remarks column is added for additional clarification. For the case of a multiple dimensioned input array, the range of IDN is given. It should be noted that a multiple dimensioned array is internally stored in a stacked column arrangement; for example, a triple array A(3,3,3) will have an IDN range from (n + 1) to (n + 27) where the A(3,1,1) element will have a corresponding (n + 3) identification number, and the A(1,1,3) element will have a corresponding (n + 19) IDN, etc.

QNAM is an alphanumeric variable used to describe the BASIC DATA variable.

STRUCTURAL DATA - Defines: Modal Matrix, Stiffness Matrix, and Damping Matrix for the rotor and flexible portion of hub. This data is provided by the Modal Service Program described in Section 10.2. All the Structural Data matrices are initialized to zero; therefore, only those elements with non-zero values require input data. Each card can contain data for as much as five elements of the matrix that belong to a common column (COL). VALU represents the data value contained in element (NROW, COL); where NROW is the row number that immediately precedes it's corresponding VALU on the data card.

TABLE 10.2

BASIC DATA IDENTIFICATION

NOTE: The range of all indices is 3.

IDN	Program Variable	Remarks
1	DTPLOT	IF DTPLOT = 0, no plots will be generated IF DTPLOT > 0, calcomp plots are scaled at DTPLOT seconds per inch.
2	DT	Iteration Time (sec). Should not be greater than 1/20 the period of the largest freq.
3	TEND	Run Duration (seconds)
4	DTPRNT	Print every DTPRNT sec.
5	NS	= 0. Shuttle not attached = 1. Shuttle attached
6	P	Number of rotor arms. 2 for T, 3 for Y.
7	HMASS	Weight (lbs) of each body. <u>Note:</u> Only the weight of the lumped masses in the first arm are read-in RDMASS(I, 1). Masses of other arms are set equal to arm #1.
8	SMASS	
9	RRMASS	
10-12	HCMASS (I)	
13-15	RDMASS (I, 1)	
16-24	IA (I, J)	Inertia (slug-ft ²) of each body. <u>Note:</u> Inertia properties of each arm is assumed identical.
25-33	IH (I, J)	
34-42	IB (I, J)	
43-69	IC (I, J, K)	
70-96	ID (I, J, K, 1)	
97-99	YH (I)	Vehicle Geometry. (feet)
100-102	RH (I)	
103-105	YB (I)	
106-108	YG (I)	
109-111	YE3 (I)	
112-114	XL (I)	
115-117	XN (I, 1)	
118-120	ZV3 (I, 1)	
201-209	YC (I, J)	
121-129	XD (I, J, 1)	
130	α_2	
131	α_3	
132-134	β	Locate shuttle (radians)

IDN	Program Variable	Remarks	
135	KBS1	Stiffness and Damping at bearing and Shuttle connection, (feet, lb, rad, sec)	
136	KBS2		
137	DELTB		
138	ETA		
139-147	KBVR (I, J)		
148-156	KBVH (I, J)		
157-165	KHS (I, J)		
166-174	KHV (I, J)		
200	SPIN	(RAD/SEC)	
210	NROT	-minimum is one for spin. } Number of flexible modes.	
211	NHUB		
212	NSYM		
213	ANTTWO		
214	PBLOCK		
		-normally set to zero.	
		-3 variables plotted per block	
220	CONTR	} CMG Variables	
221	STOP		(rad/sec)
222	START		(rad/sec)
223	TAUF		(sec)
224	H		(ft-lb-sec)
225	WNG		(rad/sec)
226	ZETG		
227	BIAS		= wobble freq. (rad/sec). + for MaxI, -for MinI. For no bias set = 0.
228	WAIT		(radians)
301-302	ZA (I)		} Initial vehicle state (feet, RAD, SEC)
370-309	PSI (I)		
310-318	UC (I, J)		
319-327	PHIC (I, J)		
331-339	UD (I, J, 1)		
340-348	PHID (I, J, 1)		
349-357	UD (I, J, 2)		
358-366	PHID (I, J, 2)		
367-375	UD (I, J, 3)		
376-384	PHID (I, J, 3)		
401-402	ZAD (I)		
404-406	WYH (I)		
407-409	PSID (I)		
410-418	UCD (I, J)		
419-427	PHICD (I, J)		
428-430	WXY (I)		
431-439	UDD (I, J, 1)		
440-448	PHIDD (I, J, 1)		
449-457	UDD (I, J, 2)		
458-466	PHIDD (I, J, 2)		
467-475	UDD (I, J, 3)		
476-484	PHIDD (I, J, 3)		
485-487	GAM (I)		
488-490	THET (I)		

There should be no blank fields between one NROW/VALU combination and another. In addition, although all the NROW/VALU fields on one card need not be filled, data must be placed on the card from left to right.

The data found on a particular card belongs to the matrix identified by IDN. The correlation between IDN and the corresponding structural data matrix, is shown in Table 10.3. It is noted that IDN will always appear in column 80. A zero in column 80 will terminate STRUCTURAL DATA read-in; hence, a blank card is required at the end of the STRUCTURAL DATA group.

For subsequent runs that do not require new structural data, a blank card is needed in place of the STRUCTURAL DATA group. If new data is desired, then an entire STRUCTURAL DATA group is required along with NRUN set to 1. There is one exception to the above outlined multiple run procedure and that is when the number of rotor modes (either total or symmetric) is changed. For this case, only new rotor modal data (PHIMR) need be read-in, provided NRUN is not set to 1.

TABLE 10.3

STRUCTURAL DATA IDENTIFICATION

<u>IDN</u>	<u>Structural Data Matrix</u>	<u>Description</u>
1	PHIMR	Rotor Modal Matrix
2	PHIMH	Hub Modal Matrix
3	KTA or KYA	Rotor Stiffness Matrix
4	CT or CY	Rotor Damping Matrix
5	KH	Hub Stiffness Matrix
6	CH	Hub Damping Matrix
7	KHG	Hub Stiffness About Point G.
8	CHG	Hub Damping About Point G.

10.1.4

SAMPLE RUN

A CMG controlled "Y" space station with a shuttle docked to its hub is the set of conditions chosen to demonstrate THP operation. This condition was chosen since it is one that yields the maximum number of print variables per print cycle.

The RUN CONTROL CARD and the BASIC DATA group is printed at the beginning of the run as shown in Table 10.4. Referring to Table 10.4, the argument of the DD array is IDN of the corresponding Basic Data variable. Any Basic Data omitted from the print indicates a zero value. For further clarification of the variable definitions refer to Tables 10.1 and 10.2.

RUN NUMBER= 1

Y CMG CONTROL

DD(1)=	0.20000E 02	DTPLOT	TIME PER INCH OF PLT
DC(2)=	0.10000E 00	DT	
DC(3)=	0.20000E 03	TEND	
DC(4)=	0.50000E 01	DTPRNT	
DD(5)=	0.10000F 01	NS	SHUTTLE ATTACHED
DD(6)=	0.30000E 01	P	NO. OF ARMS
DD(7)=	0.23000E 05	HMASS	
DC(8)=	0.31694E 06	SMASS	SHUTTLE MASS
DC(9)=	0.19000E 05	RRMASS	ROTOR RING MASS (LBS)
DD(10)=	0.45000E 04	HCMASS	
DD(11)=	0.35000E 04	HCMASS	
DD(12)=	0.12500E 06	HCMASS	
DC(13)=	0.72500E 04	RODMASS(1)	LUMPED ROTOR MASSES(LBS)
DC(14)=	0.72500E 04	RODMASS(2)	LUMPED ROTOR MASSES(LBS)
DC(15)=	0.12300E 06	RODMASS(3)	LUMPED ROTOR MASSES(LBS)
DC(16)=	0.20076E 07	IA	
DD(20)=	0.26869E 07	IA	
DD(24)=	0.68264E 06	IA	
DD(25)=	0.18038E 09	IH(1,1)	SHUTTLE INERTIA
DD(29)=	0.17877E 09	IH(2,2)	SHUTTLE INERTIA
DC(33)=	0.27250E 07	IH(3,3)	SHUTTLE INERTIA
DC(34)=	0.12692E 05	IB(1,1)	ROTOR RING INERTIA
DC(38)=	0.32781E 05	IB(2,2)	ROTOR RING INERTIA
DD(42)=	0.32781E 05	IB(3,3)	ROTOR RING INERTIA
DD(43)=	0.14160E 04	IC(1,1,1)	INERTIA OF LUMPED HUB MASS
DC(47)=	0.57580E 04	IC(2,2,1)	INERTIA OF LUMPED HUB MASS
DC(51)=	0.57580E 04	IC(3,3,1)	INERTIA OF LUMPED HUB MASS
DC(52)=	0.10050E 04	IC(1,1,2)	INERTIA OF LUMPED HUB MASS
DC(56)=	0.31200E 04	IC(2,2,2)	INERTIA OF LUMPED HUB MASS
DD(60)=	0.31200E 04	IC(3,3,2)	INERTIA OF LUMPED HUB MASS
DD(61)=	0.16441E 07	IC(1,1,3)	INERTIA OF LUMPED HUB MASS
DD(65)=	0.20349E 07	IC(2,2,3)	INERTIA OF LUMPED HUB MASS
DD(69)=	0.48576E 06	IC(3,3,3)	INERTIA OF LUMPED HUB MASS
DD(70)=	0.22731E 05	ID(1,1,1)	INERTIA OF LUMPED ROTOR MASS
DC(74)=	0.22731E 05	ID(2,2,1)	INERTIA OF LUMPED ROTOR MASS
DD(78)=	0.20830E 04	ID(3,3,1)	INERTIA OF LUMPED ROTOR MASS
DD(79)=	0.22731E 05	ID(1,1,2)	INERTIA OF LUMPED ROTOR MASS
DD(83)=	0.22731E 05	ID(2,2,2)	INERTIA OF LUMPED ROTOR MASS
DD(87)=	0.20830E 04	ID(3,3,2)	INERTIA OF LUMPED ROTOR MASS
DC(88)=	0.46548E 06	ID(1,1,3)	INERTIA OF LUMPED ROTOR MASS
DD(92)=	0.20187E 07	ID(2,2,3)	INERTIA OF LUMPED ROTOR MASS
DC(96)=	0.16426E 07	ID(3,3,3)	INERTIA OF LUMPED ROTOR MASS
DC(97)=	0.59000E 01	YH(1)	LOCATE SHUTTLE
DD(102)=	-0.12700E 03	RH(3)	LOCATE SHUTTLE
DD(103)=	-0.58150E 02	YB(1)	
DD(106)=	-0.68150E 02	YG(1)	LOCATE G
DD(109)=	0.18600E 02	YE3(1)	LOCATE E
DC(117)=	0.62500E 01	XN(3)	LOCATE ROTOR ARM - PT.N
DC(120)=	-0.32000E 02	ZV3(3)	LOCATE AR7 - PT.V
DC(123)=	0.23330E 02	XD(3,1)	LOCATE ROTOR MASSES
DD(126)=	0.57330E 02	XD(3,2)	LOCATE ROTOR MASSES
DD(129)=	0.10625E 03	XD(3,3)	LOCATE ROTOR MASSES
DD(133)=	0.15708E 01	BETA(2)	
DC(157)=	0.21193E 09	KHS(1,1)	
DC(161)=	0.21193E 09	KHS(2,2)	
DC(166)=	0.13500E 07	KHV	
DD(170)=	0.13500E 07	KHV	
DD(201)=	-0.80920E 02	YC(1,1)	LOCATE HUB MASSES
DD(204)=	-0.10015E 03	YC(1,2)	LOCATE HUB MASSES

DD(207)=	-0.13955E 03	YC(1,3) - LOCATE HUB MASSES
DC(130)=	0.20944E 01	ALPHA(2) - LOCATE ARM 2
DC(131)=	0.41887E 01	ALPHA(1) - LOCATE ARM 3
DD(135)=	0.21156E 09	KBS1 - BEARING SPRING CONSTANT
DD(137)=	0.10000E 04	DELTA - DISCONTINUITY DEFLECTION OF BEARING
DD(143)=	0.26950E 08	KBVR(2,2) - OUTER RACE DAMPING
DD(147)=	0.26950E 08	KBVR(3,3) - OUTER RACE DAMPING
DC(200)=	0.26200E 00	SPIN - NOMINAL SPIN RATE (RAD/SEC)
DC(210)=	0.60000E 01	NROT
DC(211)=	0.20000E 01	NHUB NUMBER OF HUB MODES
DC(212)=	0.30000E 01	NSYM
DC(214)=	0.40000E 01	PBLECK
DD(221)=	0.11000E-04	STOP
DC(222)=	0.15000E-04	START
DC(223)=	0.20000E 01	TAUF
DC(224)=	0.10000E 05	H- CMG WHEEL SIZE
DD(225)=	0.30000E 01	WNG- CMG NATURAL FREQ
DD(226)=	0.80000E 00	ZETG- CMG DAMPING
DD(406)=	0.59000E-03	WYH(3) DUCKING DISTURBANCE
DD(428)=	0.26200E 00	WXY(1) - INITIAL SPIN RATE (RAD/SEC)
DD(220)=	0.10000E 01	CONTR

The Structural Data matrices used during the run are part of the namelist TESTC, which is printed once at the beginning of each run. The printed output of TESTC for this sample run is not shown here because it is felt to be too lengthy for the purpose that it would serve. However, the stiffness data used was twice baseline stiffness (i.e. nominal stiffness as described in Section 4.3) with 1% structural damping for both rotor and hub.

At the end of each print cycle the variables in the following four namelists are printed:

- o TIM
- o STATE
- o CMGPR
- o SHUT

TIM - Contains only T, the problem time (in seconds).

STATE - Contains physical displacements and rates (in feet, rad, and seconds), modal displacements, rates and accelerations (i.e. XVEC, XVECD, XVECDD), and the maximum rotor ring transverse angular rate over one print cycle (i.e. WTMM in rad/sec). The latter variable is used as an indicator for stability.

CMGPR - Contains the simulation variables associated with CMG control. This namelist is printed only when there is active CMG wobble damping (i.e. CONTR \neq 0). The units are: feet, lbs, rad, sec.

SHUT - Contains the simulation variables associated with a flexibly attached shuttle. This namelist is printed only when NS \neq 0. The units are: feet, lbs, rad, sec.

As indicated by the basic data variable DTPRNT, the above namelists are printed every 5 sec. The printed output at the 100 second time point is shown in Table 10.5. As seen in table 10.5, the name of each data array is printed followed by the values presently stored in the array. Each output variable is printed in row column order; for example, the elements of a N by M array will be printed as follows: (1,1), (2,1)...(N,1) (1,2)...(N,2)...(1,M)...(N,M). For arrays that are not filled (e.g. XVEC, XVECD, and XVECDD) "garbage" is printed in place of the unused areas, and should be ignored. It should be noted that although some arrays are not filled for a particular run, these arrays may be filled during a future run.

Selected time history plots, for the above described sample run, are shown in Figure 10.26. Four plot blocks are shown (PBLOCK = 4) with three variables per block. The particular variables chosen for plotting are determined by the following statements that appear in MAIN:

```
WRITE (1)  T, WXR (2), WXR (3), WYH (2)
WRITE (1)  WYH (3), WYC (3,3) UCD (2,3)
WRITE (1)  PSID (1), UDD (1,3,2), UDD (2,3,2)
WRITE (1)  WZ (2,3,1) BDR, B
```

TABLE 10.5 THP SAMPLE OUTPUT

END

85YATE
ZAD=-0.42243119E-05,-0.81469351E-03,0.37135644E-03,ZA=-0.41771011E-04,-0.68797112E-01,0.93632527E-02,WYH=-0.309C0657E-06,
-0.35000755E-03,-0.10131979E-03,CAM=0.10882439E-03,-0.75873062E-02,0.44365600E-02,UOD=0.0
-0.1255059E-05,0.0,0.50080474E-04,-0.72405337E-05,0.0,0.19321924E-03,-0.2657580CE-04,UC=0.0
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&END
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&END

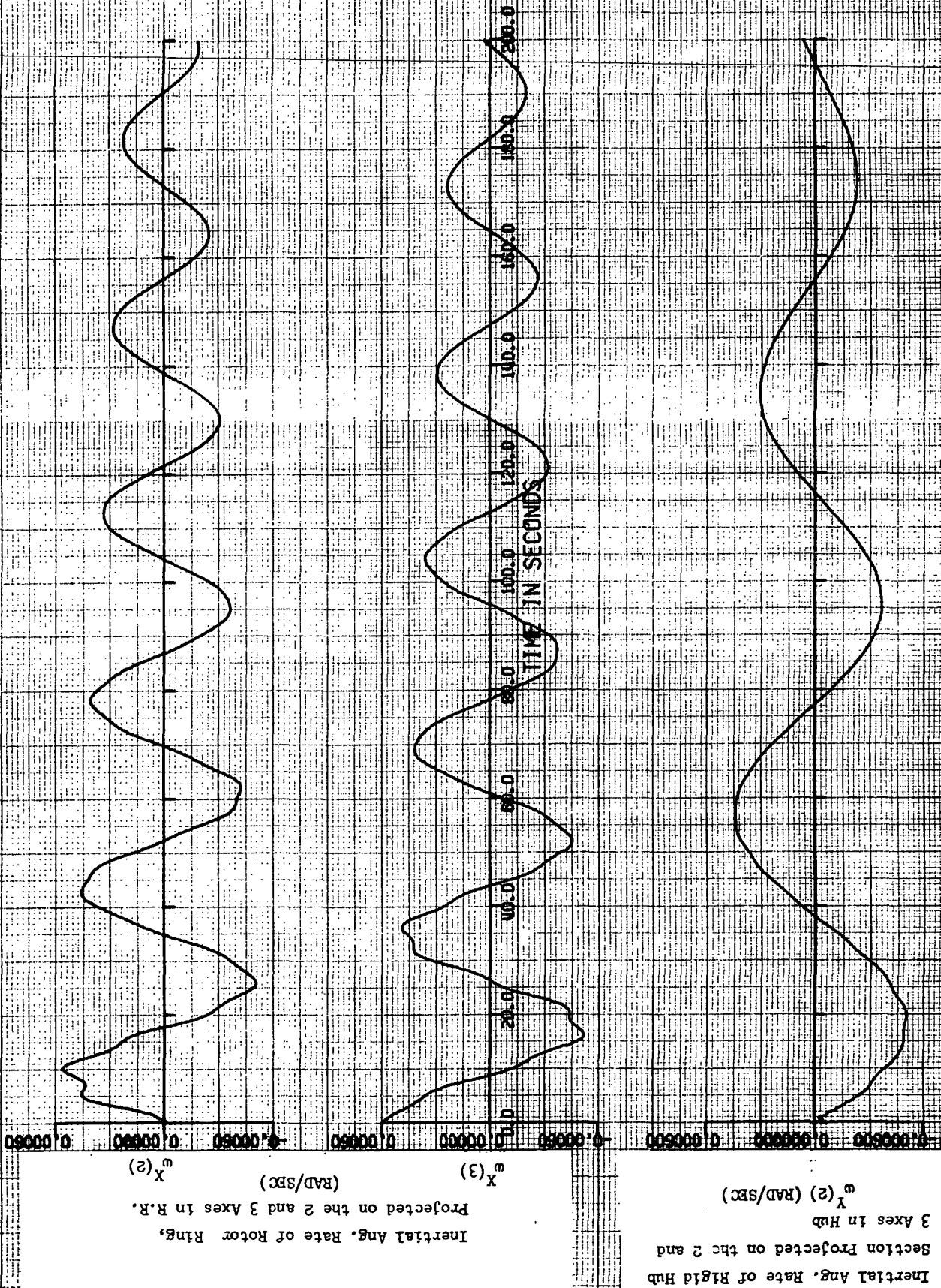
```

These plot variables are defined as follows:

- | | |
|-------------------------|--|
| T | - Time (must always appear as first variable in firstplot block) |
| WXR (2), WXR (3) | - Inertial angular rate of rotor ring projected on the two and three axes in the rotor ring. |
| WYH (2), WYH (3) | - Inertial angular rate of rigid hub section projected on the two and three axes in the rigid hub section. |
| WYC (3,3) | - Inertial angular rate of the third flexible hub mass, projected on the three axis in the third lumped mass in the hub. |
| UCD (2,3) | - The linear velocity of the third flexible hub mass relative to the rigid hub section, projected on the two axis in the third hub mass. |
| PSID (1) | - Shuttle angular rate relative to the rigid hub section, projected on the one axis in the Shuttle. |
| UDD (1,3,2), UDD(2,3,2) | - Linear velocity of the third lumped mass in the second rotor arm, projected on the mass' one and two axes. |
| WZ (2,3,1) | - Inertial angular rate of the third lumped mass in the first rotor arm, projected on the mass' two axis. |
| BDR, B | - Gimbal rate, gimbal angle. |

It should be noted that the CALCOMP subroutines that are included as part of the program package have been designed specifically for operation at the Grumman facility. Modification to these routines would most likely be required for use at another facility.

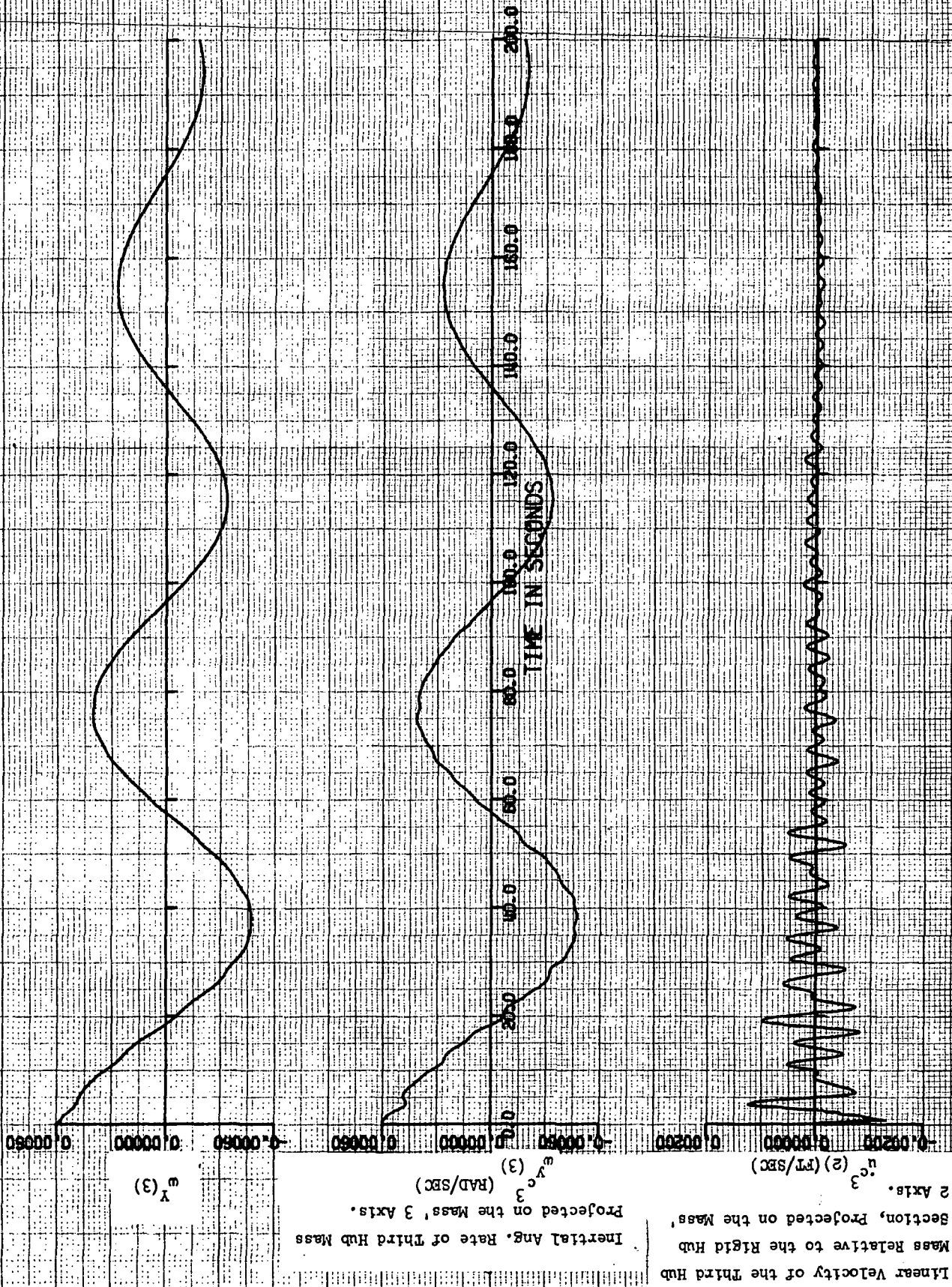
FIGURE 10.26 "Y" SHUTTLE - CMG CONTROL (H = 10,000 FT-LB-SEC)



Inertial Ang. Rate of Rigid Hub
3 Axes In Hub
Section Projected on the 2 and
 $\omega_X(2)$ (RAD/SEC)

Inertial Ang. Rate of Rotor Ring,
Projected on the 2 and 3 Axes in R.R.
 $\omega_X(3)$ (RAD/SEC)

FIGURE 10.26 (Con't)



Linear Velocity of the Third Hub Mass Relative to the Rigid Hub Section, Projected on the Mass' 2 Axis.

Mass Relative to the Rigid Hub Section, Projected on the Mass' 3 Axis.

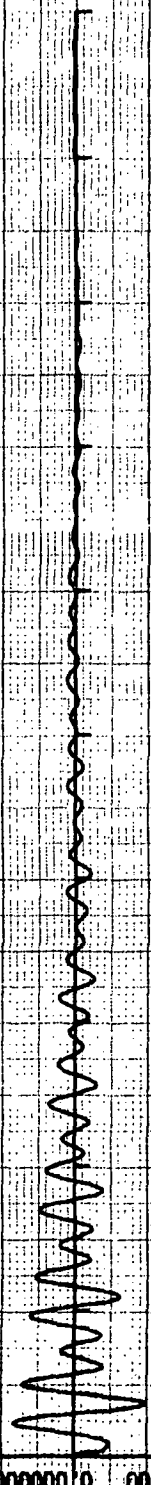
$\dot{\omega}_3$ (3)

$\dot{\omega}_3$ (3) (RAD/SEC)

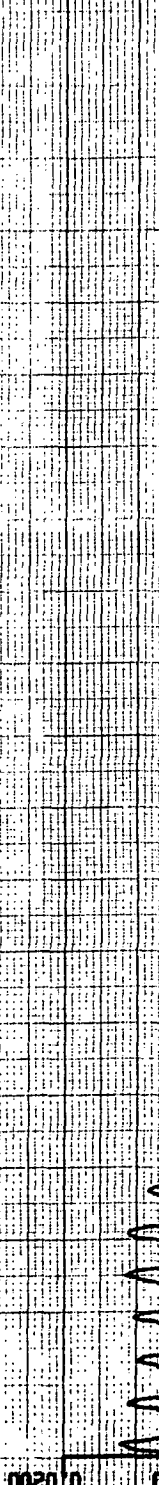
\dot{y}_3 (2) (FT/SEC)

FIGURE 10.26 (con't)

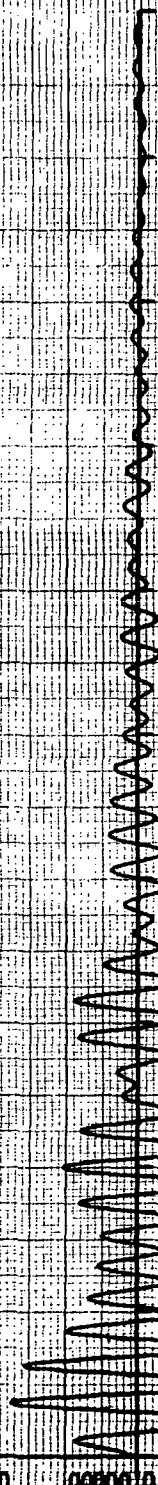
Shuttle Ang. Rate Relative to
the Hub, Projected on Shuttle
1 Axis.
 $\dot{\psi} (1)$ (RAD/SEC)



Linear Velocity of the Third Rotor
Mass in the Second Arm, Relative to the
Rotor Ring, Projected on the Mass' 1
and 2 Axes.
 $v_{23}^{(2)}$ (FT/SEC)



$v_{23}^{(1)}$ (FT/SEC)

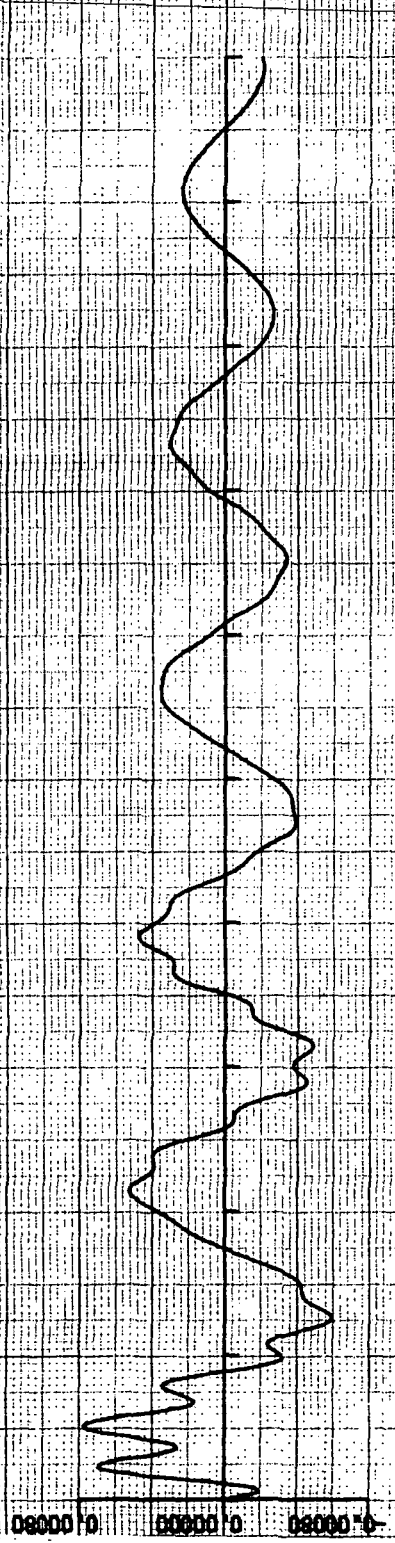


TIME IN SECONDS

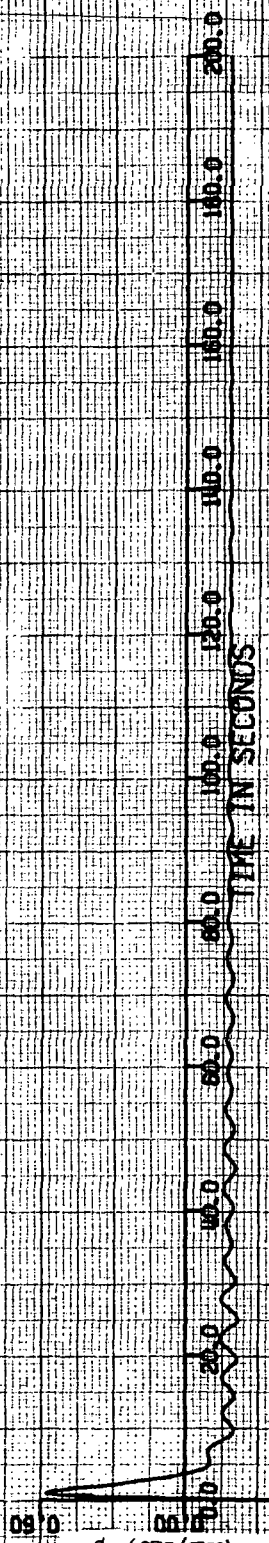
0 50.0 100.0 150.0 200.0

FIGURE 10.26 (cont.)

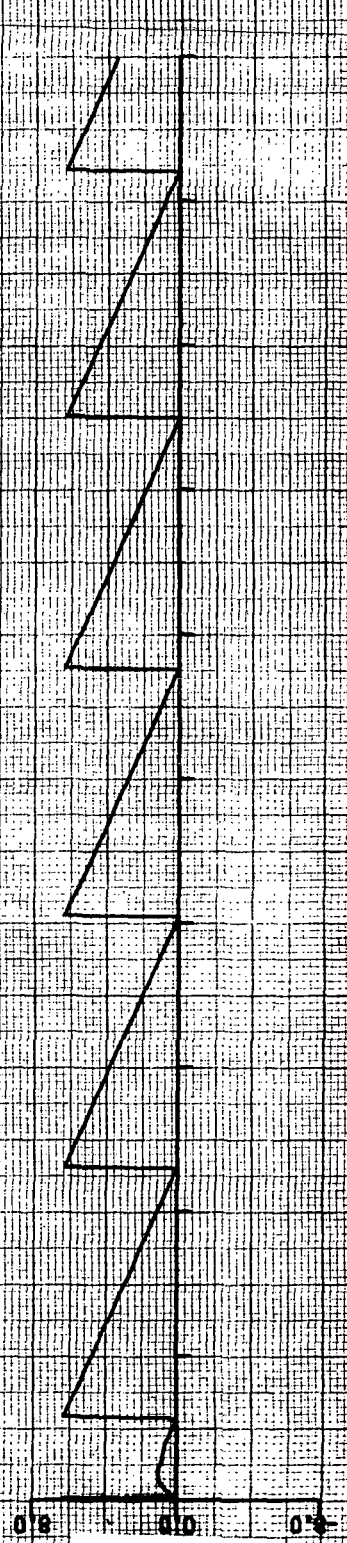
Inertial Angular Rate of the Third Rotor Mass in the First Arm, Projected on the Mass' 2 Axis. ω^2_{13} (RAD/SEC)



CMG Gimbal Rate β (RAD/SEC)



CMG Gimbal Ang. θ (RAD)



10.2 Modal Service Program

The Modal Service Program (MSP) of the General Flexible-Body Stage II digital simulation establishes the following Structural Data for use by the Time History Program:

- o Mode Shapes
- o Stiffness Matrices
- o Damping Matrices

The MSP is special purpose in that it provides the above data for only the "T" and "Y" space station configurations. The method used for formulating this data is outlined in Section 6.2. Structural data for the hub and rotor may be obtained during one program execution. Printed output of the above structural data is provided as well as punched data cards with the proper format for direct input to the Time History Program. In addition, modal frequencies are provided so as to aid the user of the Time History Program in choosing the significant mode shapes.

Operating instructions for the MSP are found in Section 10.2.2.

10.2.1 Program Description (with Listings)

As in the case of the Time History Program, program functions are accomplished by several subroutines. Figure 10.27 presents the logical flow between operational subroutines. The logical tests along with the calling of these subroutines is accomplished by the MAIN (or Executive) routine.

A brief description of each routine is presented below.

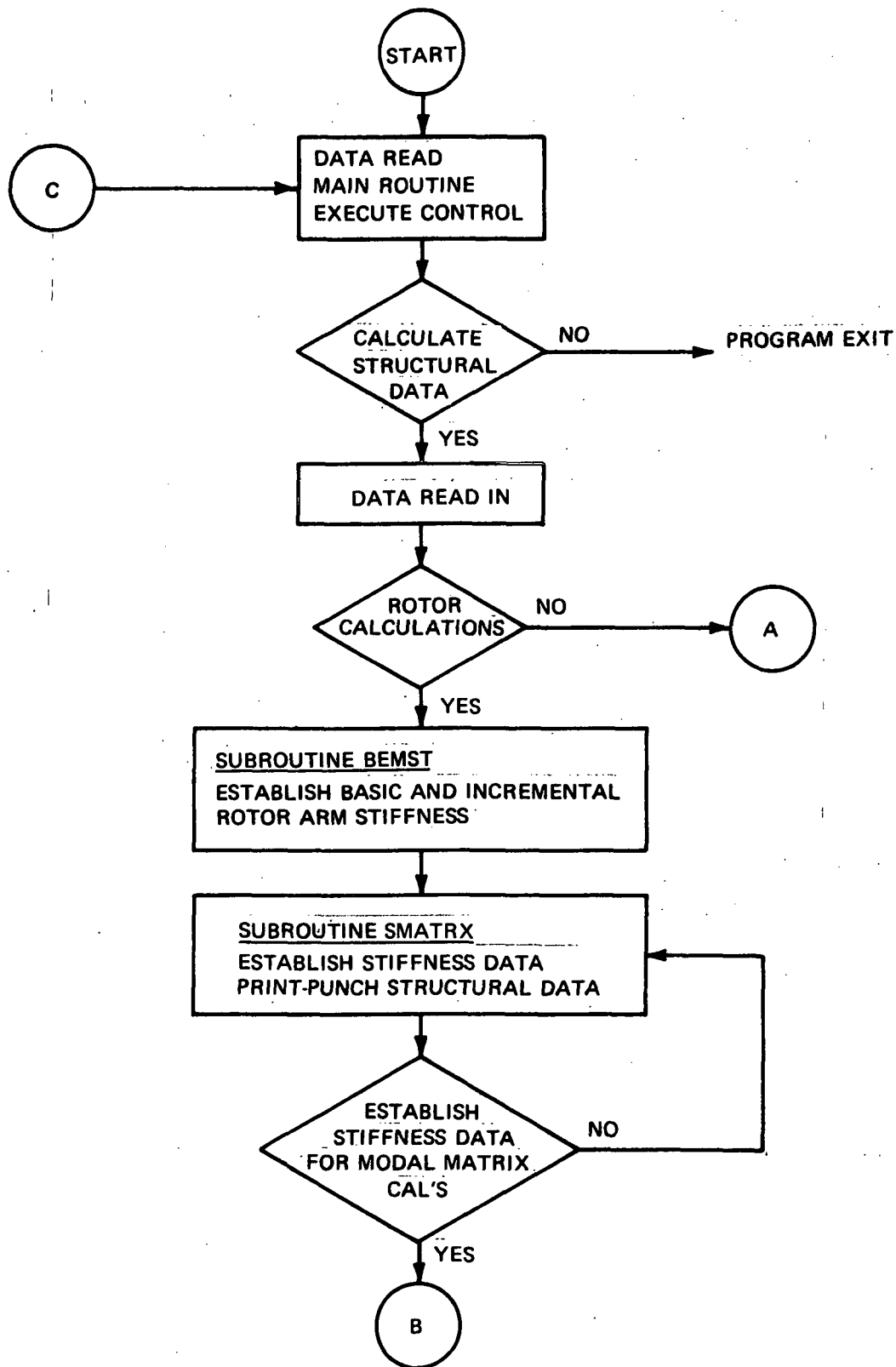


Figure 10-27. Model Service Program, Flow Chart

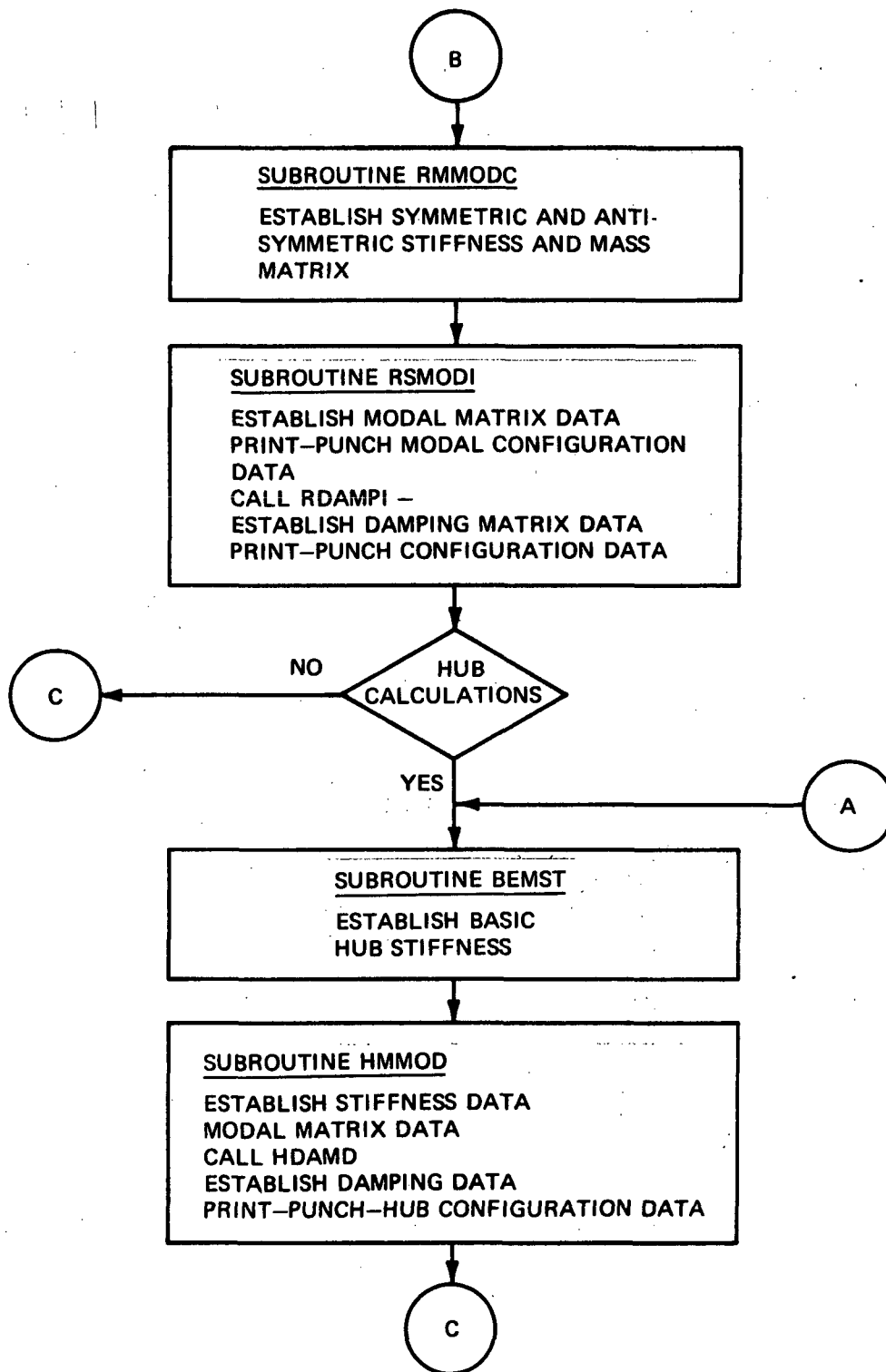


Figure 10-27. Model Service Program, Flow Chart (Cont.)

Main Routine (listing is found in Figure 10.28)

The main program accepts input data and identifies the data with the program variables. This routine establishes the calling sequence for the subroutines in which the Rotor and/or Hub stiffness, damping and modal matrices are formed.

Subroutine BEMST (figure 10.29)

Establishes an intermediate basic and incremental stiffness matrix for the rotor and an intermediate stiffness matrix for the Hub.

Subroutine SMATRIX (figure 10.30)

Establishes the final rotor stiffness matrix for use by the THP, and an intermediate stiffness matrix that is used to formulate mode shapes.

Subroutine RMODC (figure 10.31)

Establishes the rotor symmetric and antisymmetric stiffness and mass data arrays which are utilized in the computation of system modes and mode shapes.

Subroutine RSMODI (figure 10.32)

Computes the rotor modal matrix to be used by the THP.

Subroutine RDAMP (figure 10.33)

Establishes the rotor damping matrix to be used by the THP.

Subroutine HMMOD (figure 10.34)

Computes the Hub stiffness matrix and modal matrix to be used by the THP.

Subroutine HDAMP (figure 10.35)

Computes the Hub damping matrix to be used by the THP. This subroutine is called from HMMOD.

Subroutine LATE (figure 10.36)

Generates Rotor and Hub modal frequencies and mode shapes in the following manner:

The problem to be solves is,

where $[M][\ddot{\Phi}] = \lambda[K][\Phi]$ (10-5)

$$\lambda = \frac{1}{\omega^2}$$

and $[K]$ - Stiffness Matrix - (Positive Definite Symmetric)

$[M]$ - Mass Matrix (Symmetric)

$[\phi]$ - Matrix of eigen vectors (mode shapes)

$[\omega]$ - Frequency (Rad/sec)

$[\lambda]$ - Eigen value.

The first program operation is to factor the stiffness matrix $[K]$ into the form of a lower triangular matrix $[L]$ such that

$$[K] = [L] [L]^T \quad (10 - 6)$$

This above factorization is performed in subroutine "FUTILE" (figure 10.37). Subroutine "DAGGER" (figure 10.38) is then called repeatedly to form,

$$[L]^{-1} [M] [L]^{-T} = [C] \quad (10 - 7)$$

where array $[C]$ is defined by the following standard eigen value problem,

$$[C] Y = [D] Y \quad (10 - 8)$$

and $[Y] = [L]^T [\phi]$

$[D] =$ Diagonal array composed of eigen values $\lambda_1, \lambda_2, \lambda_3 \dots$

Matrix $[C]$ is then converted in subroutine "SWITCH" (figure 10.39) to a form acceptable to subroutine "SYMEIG" (figure 10.40). SYMEIG solves the standard eigen value problem. Subroutine TRIEQ (figure 10.41) is used to transform the eigenvectors to obtain $[\phi]$. Additional supporting routines required for subroutine (LATE) are:

- . TFORM (figure 10.42)
- . STURM (figure 10.43)
- . PREP (figure 10.44)
- . QSVEC (figure 10.45)
- . AND (figure 10.46)
- . DOTPRO (figure 10.47)
- . RDM (figure 10.48)

It should be noted that mode shapes will not be calculated if the stiffness matrix is not positive definite. Under these circumstances an error message "Stiffness Array not positive definite - Eigenvalues-modal vectors not computed-check input data" is printed. At this point the program run is terminated and the next data set is read.

An error message indicating a negative eigenvalue may occur. If the printed magnitude of the error indicator "CHEC" is close to zero, this error can be ignored and attributed to computer round-off

Subroutine COMPR (Figure 10.49)

Converts double dimensioned array into a column vector (or a single dimensioned array).

Subroutine ABCP (Figure 10.50)

Generates the transformation matrix for a single angle rotation.

Subroutine "Anorm" (Figure 10.51)

Normalizes each column of a matrix.

Subroutine "FWRITE" (Figure 10.52)

Generates punched data in the proper format for direct input to the THP.

Refer to section 10.1.2 for a description of the following utility subroutines:

- o WILLY
- o STACK
- o GAMMA
- o SREVN1

10.2.2 Operating Instructions

Program operation is controlled by input data cards exactly as described in Section 10.1.3 for the THP. The only difference is that the STRUCTURAL DATA block is not used. Specifically, a RUN CONTROL card is required followed by the BASIC DATA group. The required format for these data groups is identical to the THP.

ROTOR BASIC DATA

All arms are assumed to have identical geometric and mass properties. The rotor arm geometric properties that have to be considered are illustrated in Figure 10.53. Input data card identification (ID) numbers are included in Figure 10.53. Table 10.6 defines all rotor input data variables and associated ID numbers. A sample input card set-up to obtain rotor configuration data is illustrated in Table 10.7.

HUB BASIC DATA

The hub geometrical properties that must be considered along with their corresponding ID numbers are shown in Figure 10.54. Table 10.8 identifies all hub input data parameters and associated ID numbers. A sample input data set-up to obtain Hub structural data is illustrated in Table 10.9.

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```

INTEGER P,ER,EC
COMMON /PUNCH/ IRIT,TREIG,TPHFF,TSMAT,TCTYA,TK7H,TCH,TK6HG,ICHG
COMMON /BEAM1/
*HL(3),AL(3),PSS(3),HUB,EI,AE,GJ
COMMON /RHMST/
*WORK6(72,72),WORK8(72,60),WORK9(72,60),WORK11(57,57)
COMMON /RRMID/
*RRIB(3,3),DI1(3,3),DI2(3,3),DI3(3,3),DM1,DM2,DM3
COMMON /RHMST1/
*XRN(3),XRD1(3),XRD2(3),ZV(3),ALPH1(3,3),GXN(3,3),GZP(3,3),ANG(2),
*PT1,PT2,PT3,P,BKSI,TEST,QARRA(54,3),XRD3(3)
COMMON /EIGVEC/
*COLK(435),COLM(435),EIG(29,29),QMEGA1(28),QMEGA2(29),TOL1,FREQ
COMMON /HUBID/
*CI1(3,3),CI2(3,3),CI3(3,3),YHE(3),CM1,CM2,CM3,YHA(3),YHG(3),
*YHC1(3),YHC2(3),YHC3(3)
COMMON /HDAMPC/
*DAMPH
COMMON /RDAMPC/
*DAMPR
COMMON /PRINT/
*IOUT,IOIN
COMMON /TERUP/BRAK
DIMENSION DD(200)
DIMENSION QNAM(7)
DIMENSION XRP(3)
WRITE(6,901)
901 FORMAT(' TYPE FILE NUMBER TO BE USED')
READ(5,900) IOIN
900 FORMAT(I1)
IOUT=R
DO 1 I=1,200
1 DD(I)=0.
21 READ(IOIN,22) NRUN,QNAM
22 FORMAT(I3,7A4)
WRITE(IOUT,23) NRUN,QNAM
23 FORMAT(IH1,'RUN NUMBER=' ,I3,5X,7A4/)
IF(NRUN.GT.0) GO TO 25
CALL EXIT
25 READ(IOIN,26) ID,VAL,QNAM
26 FORMAT(I5,E20.5,7A4)
IF(ID.LE.0) GO TO 27
DD(ID)=VAL
WRITE(IOUT,28) ID,DD(ID),QNAM
28 FORMAT(5X,'DD(' ,I5,')=' ,G20.7,5X,7A4)
GO TO 25
27 CONTINUE
C
C INPUT DATA ARRAY
C
DO 40 I=1,3
XRD1(I)=DD(I+1)
XRD2(I)=DD(I+4)
XRD3(I)=DD(I+7)

```


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40 XRP(I)=DD(I+10)
   P=DD(14)
   I=15
   DO 29 J=1,3
   DO 29 L=1,3
   DI1(L,J)=DD(I)
29 I=I+1
   I=24
   DO 30 J=1,3
   DO 30 L=1,3
   DI2(L,J)=DD(I)
30 I=I+1
   I=33
   DO 31 J=1,3
   DO 31 L=1,3
   DI3(L,J)=DD(I)
31 I=I+1
   I=42
   DO 32 J=1,3
   DO 32 L=1,3
   RRI1(L,J)=DD(I)
32 I=I+1
   AREA=DD(51)
   AREA=AREA/144.
   AI=DD(52)
   AI=AI/(144.**2)
   GBE=DD(53)
   GBE=GBE*144.
   BE=DD(54)
   BE=BE*144.
   PMJ=DD(55)
   PMJ=PMJ/(144.**2)
   FREQ=DD(56)
   TOL1=DD(57)
   XRN(1)=DD(58)
   XRN(2)=DD(59)
   XRN(3)=DD(60)
   DM1=DD(61)
   DM2=DD(62)
   DM3=DD(63)
   PR=DD(64)
   ANG(1)=DD(65)
   ANG(2)=24.
   IF(P.EQ.2) ANG(1)=10.
   BKS1=DD(67)
   DAMPR=DD(68)

```

C
C TAPE-CARD PUNCH CONTROL-TOL. DATA
C

```

IRIT=DD(90)
TREIG=DD(91)
TPHF=DD(92)
TSMAT=DD(93)
TCTYA=DD(94)

```

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TK 7H=DD(95)
TCH=DD(96)
TK 6HG=DD(97)
TCHG=DD(98)

```

C
C
C

```

HUB INPUT DATA

```

```

HUBCAL=DD(100)
DO 35 I=1,3
YHA(I)=DD(100+I)
YHG(I)=DD(103+I)
YHC1(I)=DD(106+I)
YHC2(I)=DD(109+I)
YHC3(I)=DD(112+I)

```

```

35 YHE(I)=DD(115+I)
I=119

```

```

DO 36 J=1,3
DO 36 L=1,3
CI1(L,J)=DD(I)

```

```

36 I=I+1
I=128
DO 37 J=1,3
DO 37 L=1,3
CI2(L,J)=DD(I)

```

```

37 I=I+1
I=137
DO 38 J=1,3
DO 38 L=1,3
CI3(L,J)=DD(I)

```

```

38 I=I+1
CM1=DD(146)
CM2=DD(147)
CM3=DD(148)
DAMPH=DD(149)
AE=AREA*BE
EI=BE*AI
GJ=GBE*PMJ

```

C
C
C

```

ARM ELEMENT CALCULATIONS

```

```

AL(1)=XRD1(3)-XRN(3)
AL(2)=XRD2(3)-XRD1(3)
AL(3)=XRP(3)-XRD2(3)
ZV(1)=0.
ZV(2)=0.
ZV(3)=XRP(3)-XRD3(3)
HL(1)=YHC1(1)-YHG(1)
HL(2)=YHC2(1)-YHC1(1)
HL(3)=YHE(1)-YHC2(1)
DO 60 J=1,3

```

```

AL(J)=ABS(AL(J))
60 HL(J)=ABS(HL(J))
HUB=0.
TEST=1.

```

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C

C

C

```
PRS=PR**2
PT1=DM1*XRD1(3)*PRS
PT2=DM2*XRD2(3)*PRS
PT3=DM3*XRD3(3)*PRS
PSS(1)=PT1+PT2+PT3
PSS(2)=PT2+PT3
PSS(3)=PT3
IF(HUBCAL.EQ.1.) GO TO 45
BRAK=0.
CALL BEMST
CALL SMATRX
CALL RMMODC
CALL RSMOD1
IF(HUBCAL.EQ.0.) GO TO 50
```

C

C

C

HUB CALCULATIONS

45 CONTINUE

HUB=1.

BRAK=0.

CALL BEMST

CALL HMMOD

50 CONTINUE

IF(IRIT.LE.6) GO TO 21

IV=0

WRITE(IRIT,499) IV

499 FORMAT(78X,I2)

GO TO 21

END

IV. G. LEVEL 1, MOD 1

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```

SUBROUTINE BEMST
COMMON /BEAM1/
*HL(3),AL(3),PSS(3),HUB,EI,AE,GJ
COMMON /RHMST/
*WORK6(72,72),WORK8(72,60),WORK9(72,60),WORK11(57,57)
DIMENSION BK1(3),BK2(3),BK3(3),BK4(3),BK6(3),POL(3),SK(12,12),SSK(
*12,12),TSK(12,12),BK5(3)
DIMENSION ARY(3,3)
IF(HUB.EQ.1.) GO TO 40
DO 110 J=1,3
BK1(J)=AE/AL(J)
BK2(J)=12.*EI/AL(J)**3
BK3(J)=6.*EI/AL(J)**2
BK4(J)=GJ/AL(J)
BK5(J)=4.*EI/AL(J)
BK6(J)=2.*EI/AL(J)
110 CONTINUE
GO TO 43
40 DO 42 J=1,3
BK1(J)=AE/HL(J)
BK2(J)=12.*EI/HL(J)**3
BK3(J)=6.*EI/HL(J)**2
BK4(J)=GJ/HL(J)
BK5(J)=4.*EI/HL(J)
BK6(J)=2.*EI/HL(J)
42 CONTINUE
43 CONTINUE
DO 111 L=1,36
DO 111 J=1,12
111 WORK6(L,J)=0.
DO 112 L=1,12
DO 112 J=1,12
SK(L,J)=0.
112 SSK(L,J)=0.
C
C INCREMENTAL + BASIC STIFFNESS ARRAY CALS
C
C
C ARRAY K* CALS
C
DO 116 J=1,3
SK(1,1) =BK2(J)
SK(5,1) =BK3(J)
SK(7,1) =-BK2(J)
SK(11,1) =BK3(J)
SK(2,2) =BK2(J)
SK(4,2) =-BK3(J)
SK(8,2) =-BK2(J)
SK(10,2) =-BK3(J)
SK(3,3) =BK1(J)
SK(9,3) =-BK1(J)
SK(4,4) =BK5(J)
SK(8,4) =BK3(J)
SK(10,4) =BK6(J)

```

IV G LEVEL 1, MOD 1

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```

SK(5,5) =BK5(J)
SK(7,5) =-BK3(J)
SK(11,5) =BK6(J)
SK(6,6) =BK4(J)
SK(12,6) =-BK4(J)
SK(7,7) =BK2(J)
SK(11,7) =-BK3(J)
SK(8,8) =BK2(J)
SK(10,8) =BK3(J)
SK(9,9) =BK1(J)
SK(10,10) =BK5(J)
SK(11,11) =BK5(J)
SK(12,12) =BK4(J)
IF(HUB.EQ.1.) GO TO 80

```

C
C
C

```

ARRAY K** CALS

```

```

POL(J)=PSS(J)/(10.*AL(J))
SSK(1,1) =12.*POL(J)
SSK(5,1) =AL(J)*POL(J)
SSK(7,1) =-SSK(1,1)
SSK(11,1) =SSK(5,1)
SSK(2,2) =SSK(1,1)
SSK(4,2) =-SSK(5,1)
SSK(8,2) =-SSK(2,2)
SSK(10,2) =SSK(4,2)
SSK(4,4) =4.*AL(J)**2/3.*POL(J)
SSK(8,4) =SSK(5,1)
SSK(10,4) =-SSK(4,4)/4.
SSK(5,5) =SSK(4,4)
SSK(7,5) =SSK(4,2)
SSK(11,5) =SSK(10,4)
SSK(7,7) =SSK(1,1)
SSK(11,7) =SSK(4,2)
SSK(8,8) =SSK(1,1)
SSK(10,8) =SSK(5,1)
SSK(10,10) =SSK(4,4)
SSK(11,11) =SSK(4,4)

```

```

80 CONTINUE

```

```

KM=1
DO 114 K=1,12
DO 113 L=KM,12
SK(K,L)=SK(L,K)
SSK(K,L)=SSK(L,K)

```

```

113 CONTINUE

```

```

114 KM=KM+1

```

```

DO 115 L=1,12
DO 115 K=1,12

```

```

115 TSK(L,K)=SK(L,K)+SSK(L,K)
GO TO (101,102,103),J

```

```

101 L=1

```

```

K=1

```

```

GO TO 104

```

```

102 L=13

```

IV. G LEVEL 1. MOD 1

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K=1

GO TO 104

103 L=25

K=1

104 CALL STACK(TSK,WORK6,12,12,72,L,K)

116 CONTINUE

IF(HUB.EQ.0.) GO TO 200

DO 149 L=1,12

DO 149 J=20,31

149 WORK6(L,J)=0.

DO 150 L=1,3

DO 150 J=1,3

150 ARY(L,J)=0.

ARY(3,1)=-1.

ARY(2,2)=1.

ARY(1,3)=1.

DO 151 L=1,10,3

K=L+19

151 CALL STACK(ARY,WORK6,3,3,72,L,K)

C

C

C

N=0

152 CONTINUE

DO 153 L=1,12

LN=L+N

DO 153 J=1,12

J39=J+39

J19=J+19

WORK6(LN,J39)=0.

DO 153 K=1,12

153 WORK6(LN,J39)=WORK6(LN,J39)+WORK6(LN,K)*WORK6(K,J19)

N=N+12

IF(N.LE.24) GO TO 152

C

C

C

WORK6(1-36,40-51)=ARRAY(KSTAR*Q)

DO 154 L=1,12

L19=L+19

DO 154 J=20,31

J13=J-7

154 WORK6(J13,L19)=WORK6(L,J)

C

C

C

WORK6(13-24,20-31)=ARRAY G'(TRANSFCSE)

N=0

155 CONTINUE

DO 156 L=1,12

LN=L+N

L12=L+12

DO 156 J=1,12

J39=J+39

WORK6(LN,J)=0.

DO 156 K=1,12

IV G LEVEL 1, MOD 1

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KN=K+N

K19=K+19

156 WORK6(LN,J)=WORK6(LN,J)+WCRK6(L12,K19)*WCRK6(KN,J39)

N=N+12

IF(N.LE.24) GO TO 155

200 CONTINUE

DO 117 L=1,24

DO 117 J=25,48

117 WORK6(L,J)=0.

DO 118 L=1,12

DO 118 J=1,12

118 WORK6(L,24+J)=WORK6(L,J)

DO 119 L=7,18

DO 119 J=7,18

119 WORK6(L,24+J)=WORK6(L,24+J)+WCRK6(L+6,J-6)

DO 120 L=13,24

DO 120 J=13,24

120 WORK6(L,24+J)=WORK6(L,24+J)+WCRK6(L+12,J-12)

C

C WORK6(1-24,25-48)=ARRAY K4

C

RETURN

END

FILED SMATRIX FORTRAN P1

CALDATA TIME - SH

SUBROUTINE SMATRIX

INTEGER P,ER,EC

COMMON /PUNCH/ IRIT,TREIG,TPHHF,TSMAT,TCTYA,TK7H,TCH,TK6HG,TCHG

COMMON /BEAM1/

*HL(3),AL(3),PSS(3),HUB,EI,AE,GJ

COMMON /RHMST/

*WORK6(72,72),WORK8(72,60),WORK9(72,60),WORK11(57,57)

COMMON /RHMST1/

*XRN(3),XRD1(3),XRD2(3),ZV(3),ALPH1(3,3),GXN(3,3),GZP(3,3),ANG(2),

*PT1,PT2,PT3,P,BKS1,TEST,QARRA(54,3),XRD3(3)

COMMON /PRINT/

*IDOUT, IOIN

DIMENSION XRN(3),ZVM(3)

DO 122 L=1,24

DO 121 J=1,24

121 WORK6(L,J)=0.

122 WORK6(L,L)=1.

DO 108 L=1,3

XRN(L)=-XRN(L)

108 ZVM(L)=-ZVM(L)

CALL GAMMA(GXN,XRN)

CALL GAMMA(GZP,ZVM)

CALL STACK(GXN,WORK6,3,3,72,1,4)

CALL STACK(GZP,WORK6,3,3,72,19,22)

C

C WORK6(1-24,1-24)=ARRAY Q1

C

DO 123 L=1,24

DO 123 J=1,24

WORK6(L,J+48)=0.

DO 123 K=1,24

123 WORK6(L,J+48)=WORK6(L,J+48)+WORK6(L,K+24)*WORK6(K,J)

C

C WORK6(1-24,49-72)=ARRAY(K4*Q1)

C

DO 124 L=1,24

DO 124 J=1,24

124 WORK6(L,J+24)=WORK6(J,L)

C

C WORK6(1-24,25-48)=ARRAY Q1 (TRANSPOSE)

C

DO 125 L=1,24

DO 125 J=1,24

WORK6(L,J)=0.

DO 125 K=1,24

125 WORK6(L,J)=WORK6(L,J)+WORK6(L,K+24)*WORK6(K,J+48)

C

C WORK6(1-24,1-24)=ARRAY K5

C

WORK6(4,4)=WORK6(4,4)+PSS(1)*XRN(3)

WORK6(5,5)=WORK6(5,5)+PSS(1)*XRN(3)

WORK6(22,22)=WORK6(22,22)-PT3*ZV(3)

WORK6(23,23)=WORK6(23,23)-PT3*ZV(3)

C

C WORK6(1-24,1-24)=ARRAY K5*

FILED SMATRIX FORTRAN P1

CALldata TIME - SH

C

```

DO 220 L=1,24
DO 220 J=1,24
220 WORK11(L,J)=WORK6(L,J)
221 CONTINUE
CALL ABCP(ALPH1,ANG(1))
DO 126 L=1,24
DO 126 J=1,24
WORK6(L+24,J)=0.
126 CONTINUE
CALL STACK(ALPH1,WORK6,3,3,72,25,1)
CALL STACK(ALPH1,WORK6,3,3,72,28,4)
DO 127 L=7,24
127 WORK6(L+24,L)=1.

```

C

C

C

```

WORK6(25-48,1-24)=ARRAY Q2
DO 128 L=1,24
DO 128 J=1,24
WORK6(L+48,J)=0.
DO 128 K=1,24
128 WORK6(L+48,J)=WORK6(L+48,J)+WORK6(L,K)*WORK6(K+24,J)

```

C

C

C

```

WORK6(49-72,1-24)=ARRAY(K5*Q2)
DO 129 L=1,24
DO 129 J=1,24
129 WORK6(J+24,L+48)=WORK6(L+24,J)

```

C

C

C

```

WORK6(25-48,49-72)=ARRAY Q2(TRANPOSE)
DO 130 L=1,24
DO 130 J=1,24
WORK6(L+24,J+24)=0.
DO 130 K=1,24
130 WORK6(L+24,J+24)=WORK6(L+24,J+24)+WORK6(L+24,K+48)*WORK6(K+48,J)

```

C

C

C

```

WORK6(25-48,25-48)=ARRAY K6
IF(P.EQ.2) GO TO 145

```

C

C

C

```

THREE ARM ROTOR CALCULATIONS
CALL ABCP(ALPH1,ANG(2))
DO 131 L=1,24
DO 131 J=1,24
WORK6(L+48,J)=0.
131 CONTINUE
CALL STACK(ALPH1,WORK6,3,3,72,49,1)
CALL STACK(ALPH1,WORK6,3,3,72,52,4)
DO 132 L=7,24
132 WORK6(L+48,L)=1.

```

C

C

C

```

WORK6(49-72,1-24)=ARRAY Q2*

```

FILE0 SMATRIX FORTRAN P1

CALldata TIME - SH

DO 133 L=1,24

DO 133 J=1,24

WORK6(L+48,J+24)=0.

DO 133 K=1,24

133 WORK6(L+48,J+24)=WORK6(L+48,J+24)+WORK6(L,K)*WORK6(K+48,J)

C

C

WORK6(49-72,24-48)=ARRAY(K5*Q2*)

C

DO 134 L=1,24

DO 134 J=1,24

134 WORK6(J+24,L)=WORK6(L+48,J)

C

C

WORK6(25-48,1-24)=ARRAY Q2*(TRANSPOSE)

C

DO 135 L=1,24

DO 135 J=1,24

WORK6(L+48,J+48)=0.

DO 135 K=1,24

135 WORK6(L+48,J+48)=WORK6(L+48,J+48)+WORK6(L+24,K)*WORK6(K+48,J+24)

C

C

WORK6(49-72,49-72)=ARRAY K6*

C

DO 136 L=1,24

DO 136 J=1,24

WORK6(L+24,J)=0.

WORK6(L+48,J)=0.

WORK6(L,J+24)=0.

WORK6(L,J+48)=0.

WORK6(L+48,J+24)=0.

136 WORK6(L+24,J+48)=0.

C

C

WORK6(1-72,1-72)=ARRAY K7Y

C

DO 137 L=1,72

DO 137 J=1,60

137 WORK8(L,J)=0.

DO 138 L=1,24

138 WORK8(L,L)=1.

DO 139 L=1,18

WORK8(L+30,L+24)=1.

139 WORK8(L+54,L+42)=1.

DO 140 L=1,6

WORK8(L+24,L)=1.

140 WORK8(L+48,L)=1.

C

C

WORK8=ARRAY Q3Y

C

DO 141 L=1,72

DO 141 J=1,60

WORK9(L,J)=0.

DO 141 K=1,72

141 WORK9(L,J)=WORK9(L,J)+WORK6(L,K)*WORK8(K,J)

DO 142 L=1,72

DO 142 J=1,60

142 WORK6(J,L)=WORK8(L,J)

FILE0 SMATRX FORTRAN P1

CALldata TIME - SHA

```

C
C   WORK6(1-60,1-72)=ARRAY Q3Y(TRANPOSE)
C
  DO 143 L=1,60
  DO 143 J=1,60
  WORK8(L,J)=0.
  DO 143 K=1,72
143  WORK8(L,J)=WORK8(L,J)+WORK6(L,K)*WORK9(K,J)
C
C   WORK8(1-60,1-60)=ARRAY K8Y
C
  DO 144 L=1,57
  DO 144 J=1,57
144  WORK9(L,J)=WORK8(L+3,J+3)
  WORK9(2,2)=WORK9(2,2)+BKSI
  WORK9(3,3)=WORK9(3,3)+BKSI
  DO 240 L=58,72
  DO 240 J=1,60
240  WORK9(L,J)=0.
  DO 241 L=1,57
  DO 241 J=58,60
241  WORK9(L,J)=0.
C
C   WORK9(1-57,1-57)=ARRAY K10Y
C
  GO TO 155
C
C   TWO ARM ROTOR CALCULATIONS
C
145  DO 146 L=1,24
  DO 146 J=1,24
  WORK6(L+24,J)=0.
146  WORK6(L,J+24)=0.
C
C   WORK6(1-48,1-48)=ARRAY K7T
C
  DO 147 L=1,48
  DO 147 J=1,42
147  WORK8(L,J)=0.
  DO 148 L=1,24
148  WORK8(L,L)=1.
  DO 149 L=1,18
149  WORK8(L+30,L+24)=1.
  DO 150 L=1,6
150  WORK8(L+24,L)=1.
C
C   WORK8(1-48,1-42)=ARRAY Q3T
C
  DO 151 L=1,48
  DO 151 J=1,42
  WORK9(L,J)=0.
  DO 151 K=1,48
151  WORK9(L,J)=WORK9(L,J)+WORK6(L,K)*WORK8(K,J)
C
C   WORK9(1-48,1-42)=ARRAY(K7T*Q3T)

```

FILE0 SMATRX FORTRAN P1

CALLDATA TIME SH

C

DO 152 L=1,48

DO 152 J=1,42

152 WORK6(J,L)=WORK8(L,J)

C

C

WORK6(1-42,1-48)=ARRAY Q3T(TRANPOSE)

C

DO 153 L=1,42

DO 153 J=1,42

WORK8(L,J)=0.

DO 153 K=1,48

153 WORK8(L,J)=WORK8(L,J)+WORK6(L,K)*WORK9(K,J)

C

C

WORK8(1-42,1-42)=ARRAY K8T

C

DO 154 L=1,39

DO 154 J=1,39

154 WORK9(L,J)=WORK8(L+3,J+3)

WORK9(2,2)=WORK9(2,2)+BKSI

WORK9(3,3)=WORK9(3,3)+BKSI

DO 230 L=40,72

DO 230 J=1,60

230 WORK9(L,J)=0.

DO 231 L=1,39

DO 231 J=40,60

231 WORK9(L,J)=0.

C

C

WORK9(1-39,1-39)=ARRAY K10T

C

155 IF(TEST.EQ.2) GO TO 160

WORK9(2,2)=WORK9(2,2)-BKSI

WORK9(3,3)=WORK9(3,3)-BKSI

C

C

WORK9(1-39,1-39)=ARRAY K9TA 2-ARM ROTCR

C

WORK9(1-57,1-57)=ARRAY K9YA 3-ARM RCTCR

C

DO 156 L=1,24

DO 156 J=1,24

156 WORK6(L,J)=WORK11(L,J)

WORK6(2,2)=WORK6(2,2)-PT1/XRD1(3)-PT2/XRD2(3)-PT3/XRD3(3)

WORK6(2,8)=WORK6(2,8)+PT1/XRD1(3)

WORK6(8,2)=WORK6(8,2)+PT1/XRD1(3)

WORK6(8,8)=WORK6(8,8)-PT1/XRD1(3)

WORK6(2,14)=WORK6(2,14)+PT2/XRD2(3)

WORK6(14,2)=WORK6(14,2)+PT2/XRD2(3)

WORK6(14,14)=WORK6(14,14)-PT2/XRD2(3)

WORK6(2,20)=WORK6(2,20)+PT3/XRD3(3)

WORK6(20,2)=WORK6(20,2)+PT3/XRD3(3)

WORK6(20,20)=WORK6(20,20)-PT3/XRD3(3)

TEST=2.

C

C

WORK6(1-24,1-24)=ARRAY K5**

C

C

STIFFNESS MATRIX CALCULATIONS

C

C

```

FILED SMATRX  FORTRAN  P1                                CALldata  TIME - SH/

      DO 158 L=1,57
      DO 157 J=1,57
157  WORK11(L,J)=0.
158  WORK11(L,L)=1.
      CALL GAMMA(GXN,XRD1)
      CALL STACK(GXN,WORK11,3,3,57,4,1)
      DO 259 L=1,3
      WORK11(L+6,L)=-1.
      WORK11(L+12,L)=-1.
      WORK11(L+18,L)=-1.
259  CONTINUE
      CALL GAMMA(GXN,XRD2)
      CALL STACK(GXN,WORK11,3,3,57,10,1)
      CALL GAMMA(GXN,XRD3)
      CALL STACK(GXN,WORK11,3,3,57,16,1)

C
C  WORK11(4-21,1-3)=ARRAY Q4
C
      CALL ABCP(ALPH1,ANG(1))
      DO 159 L=1,18
      DO 159 J=1,3
      WORK11(L+21,J)=0.
      DO 159 K=1,3
159  WORK11(L+21,J)=WORK11(L+21,J)+WORK11(L+3,K)*ALPH1(K,J)

C
C  WORK11(22-39,1-3)=ARRAY Q5
C
      DO 235 L=1,36
      DO 235 J=1,3
235  QARRA(L,J)=WORK11(L+3,J)
      IF(P.EQ.2) GO TO 186

C
C  THREE ARM ROTOR CALCULATIONS
C
      CALL ABCP(ALPH1,ANG(2))
      DO 181 L=1,18
      DO 181 J=1,3
      WORK11(L+39,J)=0.
      DO 181 K=1,3
181  WORK11(L+39,J)=WORK11(L+39,J)+WORK11(L+3,K)*ALPH1(K,J)

C
C  WORK11(40-57,1-3)=ARRAY Q5*
C
      DO 236 L=1,54
      DO 236 J=1,3
236  QARRA(L,J)=WORK11(L+3,J)
      DO 182 L=4,57
      DO 182 J=1,3
182  WORK11(L,J)=-WORK11(L,J)

C
C  WORK11(4-57,1-3)=-ARRAY(Q4,Q5,Q5*)
C  WORK11(1-57,1-57)=ARRAY QY7
C
      DO 183 L=1,57
      DO 183 J=1,57

```

FILE0 SMATRX FORTRAN P1

CALldata TIME - SF

WORK8(L,J)=0.

DO 183 K=1,57

183 WORK8(L,J)=WORK8(L,J)+WORK9(L,K)*WORK11(K,J)

C

C

WORK8(1-57,1-57)=ARRAY((K11Y*)QY7)

C

DO 184 L=1,57

DO 184 J=1,57

184 WORK11(L,J)=WORK8(L,J)

C

C

WORK11(1-57,1-57)=ARRAY KYA

C

GO TO 191

C

C

TWO ARM ROTOR CALCULATIONS

C

186 DO 187 L=4,39

DO 187 J=1,3

187 WORK11(L,J)=-WORK11(L,J)

C

C

WORK11(4-39,1-3)=-ARRAY(Q4,Q5)

C

WORK11(1-39,1-39)=ARRAY QT7

C

DO 188 L=1,39

DO 188 J=1,39

WORK8(L,J)=0.

DO 188 K=1,39

188 WORK8(L,J)=WORK8(L,J)+WORK9(L,K)*WORK11(K,J)

C

C

WORK8(1-39,1-39)=ARRAY(K11T*)*QT7)

C

DO 189 L=1,39

DO 189 J=1,39

189 WORK11(L,J)=WORK8(L,J)

C

C

WORK11(1-39,1-39)=ARRAY KTA

C

C

STIFFNESS MATRIX PRINTOUT

C

191 IF(P.EQ.2) GO TO 194

WRITE(IOUT,192)

192 FORMAT('0','THREE ARM ROTOR STIFFNESS MATRIX')

ER=57

EC=57

GO TO 196

194 WRITE(IOUT,195)

195 FORMAT('0','TWO ARM ROTOR STIFFNESS MATRIX')

ER=39

EC=39

196 CONTINUE

DO 279 L=1,ER

279 WORK11(L,1)=0.

K=53

IF(P.EQ.2) K=35

FILED SMATRX FORTRAN P1

CALldata TIME - SH

```
L=5
M=17
280 CONTINUE
   WORK11(L,1)=-PT1
   WORK11(L+6,1)=-PT2
   WORK11(M,1)=-PT3
   M=M+18
   L=L+18
   IF(M.LE.K) GO TO 280
   CALL WILLY('S-MATRIX',WORK11,57,57,1,ER,1,EC)
   CALL PWRIT(WORK11,57,57,1,ER,1,EC,3,TSMAT,IRIT)
   GO TO 221
160 RETURN
END
```

FILED RMMODC FORTRAN P1

CALldata TIME - SH

```

SUBROUTINE RMMODC
  INTEGER P,ER,EC
  COMMON /RHMST/
  *WORK6(72,72),WORK8(72,60),WORK9(72,60),WORK11(57,57)
  COMMON /RHMST1/
  *XRN(3),XRD1(3),XRD2(3),ZV(3),ALPH1(3,3),GXN(3,3),GZP(3,3),ANG(2),
  *PT1,PT2,PT3,P,BKS1,TEST,QARRA(54,3),XRD3(3)
  COMMON /RRMID/
  *RRIB(3,3),DI1(3,3),DI2(3,3),DI3(3,3),DM1,DM2,DM3
  COMMON /EIGVEC/
  *COLK(435),COLM(435),EIG(29,29),QMEGA1(28),QMEGA2(29),TCL1,FREQ
  COMMON /PRINT/
  *IOUT,IOIN

```

C
C
C

ROTOR MASS MATRIX CALCULATIONS

```

DO 161 L=1,57
DO 161 J=1,57
161 WORK8(L,J)=0.
DO 163 L=1,3
DO 162 J=1,3
WORK8(L,J)=RRIB(L,J)
WORK8(L+6,J+6)=DI1(L,J)
WORK8(L+12,J+12)=DI2(L,J)
WORK8(L+18,J+18)=DI3(L,J)
WORK8(L+24,J+24)=DI1(L,J)
WORK8(L+30,J+30)=DI2(L,J)
162 WORK8(L+36,J+36)=DI3(L,J)
WORK8(L+3,L+3)=DM1
WORK8(L+9,L+9)=DM2
WORK8(L+15,L+15)=DM3
WORK8(L+21,L+21)=DM1
WORK8(L+27,L+27)=DM2
163 WORK8(L+33,L+33)=DM3

```

C
C
C

WORK8(1-39,1-39)=ARRAY M2T

IF(P.EQ.2) GO TO 200

C
C
C

THREE ROTOR SYMMETRIC AND ANTISYMMETRIC MOTION CALCULATIONS

```

DO 165 L=1,3
DO 164 J=1,3
WORK8(L+42,J+42)=DI1(L,J)
WORK8(L+48,J+48)=DI2(L,J)
164 WORK8(L+54,J+54)=DI3(L,J)
WORK8(L+39,L+39)=DM1
WORK8(L+45,L+45)=DM2
165 WORK8(L+51,L+51)=DM3

```

C
C
C

WORK8(1-57,1-57)=ARRAY M2Y

CALL RDAMP

DO 166 L=1,57

DO 166 J=1,57


```

FILE0 RMMODC   FORTRAN  P1                               CALldata  TIME - SH
166 WORK6(L,J)=0.
   WORK6(2,1)=1.
   WORK6(1,29)=1.
   WORK6(3,30)=1.
   L=4
   DO 167 J=1,9
   WORK6(L,J+1)=1.
   WORK6(L+1,J+30)=1.
167 L=L+2
   DO 168 L=1,18
   WORK6(L+21,L+10)=1.
168 WORK6(L+21,L+39)=1.
   DO 169 L=1,18,2
   WORK6(L+39,L+10)=1.
   WORK6(L+40,L+11)=-1.
   WORK6(L+39,L+39)=-1.
169 WORK6(L+40,L+40)=1.
C
C   WORK6(1-57,1-28)=ARRAY YSYM2
C   WORK6(1-57,29-57)=ARRAY YANT2
C
   DO 170 L=1,57
   DO 170 J=1,28
   WORK11(L,J)=0.
   DO 170 K=1,57
170 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK6(K,J)
   DO 171 L=1,57
   DO 171 J=29,57
   WORK11(L,J)=0.
   DO 171 K=1,57
171 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK6(K,J)
C
C   WORK11(1-57,1-28)=ARRAY(K10Y*YSYM2)
C   WORK11(1-57,29-57)=ARRAY(K10Y*YANT2)
C
   DO 172 L=1,57
   DO 172 J=1,28
   WORK9(L,J)=0.
   DO 172 K=1,57
172 WORK9(L,J)=WORK9(L,J)+WORK8(L,K)*WORK6(K,J)
   DO 173 L=1,57
   DO 173 J=29,57
   WORK9(L,J)=0.
   DO 173 K=1,57
173 WORK9(L,J)=WORK9(L,J)+WORK8(L,K)*WORK6(K,J)
C
C   WORK9(1-57,1-28)=ARRAY(M2Y*YSYM2)
C   WORK9(1-57,29-57)=ARRAY(M2Y*YANT2)
C
   DO 174 L=1,57
   DO 174 J=1,57
174 WORK8(J,L)=WORK6(L,J)
C
C   WORK8(1-28,1-57)=ARRAY YSYM2(TRANSP0SE)
C   WORK8(29-57,1-57)=ARRAY YANT2(TRANSP0SE)

```

FILE0 RMMODC FORTRAN P1

CALldata TIME - SHA

C

```

DO 175 L=1,72
DO 175 J=1,72
175 WORK6(L,J)=0.
DO 176 L=1,28
DO 176 J=1,28
WORK6(L,J)=0.
DO 176 K=1,57
176 WORK6(L,J)=WORK6(L,J)+WORK8(L,K)*WORK11(K,J)
DO 177 L=1,28
DO 177 J=1,28
WORK6(L,J+39)=0.
DO 177 K=1,57
177 WORK6(L,J+39)=WORK6(L,J+39)+WORK8(L,K)*WORK9(K,J)

```

C

```

C WORK6(1-28,1-28)=ARRAY KYSYM
C WORK6(1-28,40-67)=ARRAY MYSYM
C

```

C

```

DO 178 L=1,29
DO 178 J=1,29
WORK6(L+39,J)=0.
DO 178 K=1,57
178 WORK6(L+39,J)=WORK6(L+39,J)+WORK8(L+28,K)*WORK11(K,J+28)
DO 179 L=1,29
DO 179 J=1,29
WORK6(L+39,J+39)=0.
DO 179 K=1,57
179 WORK6(L+39,J+39)=WORK6(L+39,J+39)+WORK8(L+28,K)*WORK9(K,J+28)

```

C

```

C WORK6(40-68,1-29)=ARRAY KYANT
C WORK6(40-68,40-68)=ARRAY MYANT
C

```

C

```

DO 180 L=1,57
DO 180 J=1,57
180 WORK11(J,L)=WORK8(L,J)

```

C

```

C WORK11(1-57,1-28)=ARRAY YSYM2
C WORK11(1-57,29-57)=ARRAY YANT2
C

```

C

```

GO TO 250

```

C

```

C TWO ROTOR SYMMETRIC AND ANTISYMMETRIC MOTION CALCULATIONS
C

```

C

```

200 CONTINUE
CALL RDAMP
DO 201 L=1,39
DO 201 J=1,39
201 WORK6(L,J)=0.
WORK6(3,1)=1.
WORK6(1,20)=1.
WORK6(2,21)=1.
DO 202 L=1,18
WORK6(L+3,L+1)=1.
202 WORK6(L+3,L+21)=1.
DO 203 L=1,18,2

```

FILE0 RMODC FORTRAN P1

CALldata TIME - SH

```

WORK6(L+21,L+1)=1.
WORK6(L+22,L+2)=-1.
WORK6(L+21,L+21)=-1.

```

```

203 WORK6(L+22,L+22)=1.

```

```

C
C
C
C

```

```

WORK6(1-39,1-19)=ARRAY TSYM2
WORK6(1-39,20-39)=ARRAY TANT2

```

```

DO 204 L=1,39
DO 204 J=1,19
WORK11(L,J)=0.
DO 204 K=1,39

```

```

204 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK6(K,J)

```

```

DO 205 L=1,39
DO 205 J=20,39
WORK11(L,J)=0.
DO 205 K=1,39

```

```

205 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK6(K,J)

```

```

C
C
C
C

```

```

WORK11(1-39,1-19)=ARRAY(K10T*TSYM2)
WORK11(1-39,20-39)=ARRAY(K10T*TANT2)

```

```

DO 206 L=1,39
DO 206 J=1,19
WORK9(L,J)=0.
DO 206 K=1,39

```

```

206 WORK9(L,J)=WORK9(L,J)+WORK8(L,K)*WORK6(K,J)

```

```

DO 207 L=1,39
DO 207 J=20,39
WORK9(L,J)=0.
DO 207 K=1,39

```

```

207 WORK9(L,J)=WORK9(L,J)+WORK8(L,K)*WORK6(K,J)

```

```

C
C
C
C

```

```

WORK9(1-39,1-19)=ARRAY(M2T*TSYM2)
WORK9(1-39,20-39)=ARRAY(M2T*TANT2)

```

```

DO 208 L=1,39
DO 208 J=1,39

```

```

208 WORK8(J,L)=WORK6(L,J)

```

```

C
C
C
C

```

```

WORK8(1-19,1-39)=ARRAY TSYM2(TRANSP0SE)
WORK8(20-39,1-39)=ARRAY TANT2(TRANSP0SE)

```

```

DO 209 L=1,72
DO 209 J=1,72
209 WORK6(L,J)=0.

```

```

DO 210 L=1,19
DO 210 J=1,19
WORK6(L,J)=0.
DO 210 K=1,39

```

```

210 WORK6(L,J)=WORK6(L,J)+WORK8(L,K)*WORK11(K,J)

```

```

DO 211 L=1,19
DO 211 J=1,19
WORK6(L,J+39)=0.
DO 211 K=1,39

```

FILE0 RMMODC FORTRAN P1

CALLDATA TIME - SH/

211 WORK6(L,J+39)=WORK6(L,J+39)+WORK8(L,K)*WCRK9(K,J)

C

C

WORK6(1-19,1-19)=ARRAY KTSYM

C

WORK6(1-19,40-58)=ARRAY MTSYM

C

DO 212 L=1,20

DO 212 J=1,20

WORK6(L+39,J)=0.

DO 212 K=1,39

212 WORK6(L+39,J)=WORK6(L+39,J)+WORK8(L+19,K)*WCRK11(K,J+19)

DO 213 L=1,20

DO 213 J=1,20

WORK6(L+39,J+39)=0.

DO 213 K=1,39

213 WORK6(L+39,J+39)=WORK6(L+39,J+39)+WORK8(L+19,K)*WCRK9(K,J+19)

C

C

WORK6(40-59,1-20)=ARRAY KTANT

C

WORK6(40-59,40-59)=ARRAY MTANT

C

DO 214 L=1,39

DO 214 J=1,39

214 WORK11(J,L)=WORK8(L,J)

C

C

WORK11(1-39,1-19)=ARRAY TSYM2

C

WORK11(1-39,20-39)=ARRAY TANT2

C

250 CONTINUE

RETURN

END

IV G LEVEL 1, MOD 1

RSMOD1

DATE = 71363

13.18.49

```

SUBROUTINE RSMOD1
INTEGER P, ER, EC
COMMON /PUNCH/ IRIT, TREIG, TPHHP, TSMAT, TCTYA, TK7H, TCH, TK6HG, TCHG
COMMON /RHMST/
*WORK6(72,72), WORK8(72,60), WORK9(72,60), WORK11(57,57)
COMMON /RHMST1/
*XRN(3), XRD1(3), XRD2(3), ZV(3), ALPH1(3,3), GXN(3,3), GZP(3,3), ANG(2),
*PT1, PT2, PT3, P, BKS1, TEST, QARRA(54,3), XRD3(3)
COMMON /RRMID/
*RBIB(3,3), DI1(3,3), DI2(3,3), DI3(3,3), DM1, DM2, DM3
COMMON /EIGVEC/
*COLK(435), COLM(435), EIG(29,29), QMEGA1(28), QMEGA2(29), TOL1, FREQ
COMMON /PRINT/
*IOUT, IOIN
COMMON /TERUP/ BRAK
RAD2=6.2831853
QMEG=RAD2*PREQ
QMEG2=QMEG**2
IP(P.EQ.2) GO TO 260

```

```

C
C   THREE ARM ROTOR K-BAR CALCULATIONS
C   FOR ANTI-SYMMETRIC PROBLEM ONLY
C

```

```

DO 255 L=1,29
DO 255 J=1,29
255 WORK6(L+39,J)=WORK6(L+39,J)+QMEG2*WORK6(L+39,J+39)

```

```

C
C   WORK6(40-68,1-29)=ARRAY KYANT-BAR
C
GO TO 265

```

```

C
C   TWO ARM ROTOR-K-BAR CALCULATIONS
C   FOR ANTI-SYMMETRIC PROBLEM ONLY
C

```

```

260 DO 261 L=1,20
DO 261 J=1,20
261 WORK6(L+39,J)=WORK6(L+39,J)+QMEG2*WORK6(L+39,J+39)

```

```

C
C   WORK6(40-59,1-20)=ARRAY KTANT-BAR
C
GO TO 270

```

```

C
C   THREE ARM ROTOR EIGENVECTOR-EIGENVALUE CALCULATIONS
C

```

```

C   SYMMETRIC PROBLEM CALCULATIONS
C

```

```

265 CALL COMPR(WORK6,COLK,72,1,1,28)
CALL COMPR(WORK6,COLM,72,1,40,28)
CALL LATE(28,COLK,28,TOL1,QMEGA1,NIX1,COLM,WORK6,72,1,QMEG2,EIG)
IF(BRAK.EQ.1.) RETURN

```

```

C
C   WORK6(1-28,1-28)=ARRAY EIG-YSYM1
C

```

```

C   ANTI-SYMMETRIC PROBLEM CALCULATIONS

```

IV G LEVEL 1, MOD 1

RSMOD1

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```

C
CALL COMPR(WORK6, COLK, 72, 40, 1, 29)
CALL COMPR(WORK6, COLM, 72, 40, 40, 29)
CALL LATE(29, COLK, 29, TOL1, QMEGA2, NIX2, COLM, WORK6, 72, 40, QMEG2, EIG)
IF(BRAK.EQ.1.) RETURN

C
C
C
WORK6(40-68, 1-29) = ARRAY EIG-YANT1

DO 266 L=1, 57
DO 266 J=1, 28
WORK8(L, J) = 0.
DO 266 K=1, 28
266 WORK8(L, J) = WORK8(L, J) + WORK11(L, K) * WORK6(K, J)
DO 267 L=1, 57
DO 267 J=1, 29
WORK8(L, J+28) = 0.
DO 267 K=29, 57
267 WORK8(L, J+28) = WORK8(L, J+28) + WORK11(L, K) * WORK6(K+11, J)

C
C
C
WORK8(1-57, 1-28) = ARRAY EIG-YSYM2
WORK8(1-57, 29-57) = ARRAY EIG-YANT2

C
C
C
GO TO 275

C
C
C
TWO ARM ROTOR EIGENVECTOR-EIGENVALUE CALCULATIONS

C
C
C
SYMMETRIC PROBLEM CALCULATIONS

270 CALL COMPR(WORK6, COLK, 72, 1, 1, 19)
CALL COMPR(WORK6, COLM, 72, 1, 40, 19)
CALL LATE(19, COLK, 19, TOL1, QMEGA1, NIX1, COLM, WORK6, 72, 1, QMEG2, EIG)
IF(BRAK.EQ.1.) RETURN

C
C
C
C
C
C
WORK6(1-19, 1-19) = ARRAY EIG-TSYM1

C
C
C
ANTI-SYMMETRIC PROBLEM CALCULATIONS

CALL COMPR(WORK6, COLK, 72, 40, 1, 20)
CALL COMPR(WORK6, COLM, 72, 40, 40, 20)
CALL LATE(20, COLK, 20, TOL1, QMEGA2, NIX2, COLM, WORK6, 72, 40, QMEG2, EIG)
IF(BRAK.EQ.1.) RETURN

C
C
C
C
C
C
WORK6(40-59, 1-20) = ARRAY EIG-TANT1

DO 271 L=1, 39
DO 271 J=1, 19
WORK8(L, J) = 0.
DO 271 K=1, 19
271 WORK8(L, J) = WORK8(L, J) + WORK11(L, K) * WORK6(K, J)
DO 272 L=1, 39
DO 272 J=1, 20
WORK8(L, J+19) = 0.
DO 272 K=20, 39
272 WORK8(L, J+19) = WORK8(L, J+19) + WORK11(L, K) * WORK6(K+20, J)

```

IV G LEVEL 1, MOD 1

RSMOD1

DATE = 71363

13.18.49

```

C
C   WORK8(1-39,1-19)=ARRAY EIG-TSYM2
C   WORK8(1-39,20-39)=ARRAY EIG-TANT2
275 CONTINUE
    DO 279 L=1,57
    DO 276 J=1,57
276 WORK9(L,J)=0.
279 WORK9(L,L)=1.
    J=29
    K=54
    IF(P.EQ.3) GO TO 400
    J=20
    K=36
400 CONTINUE
    DO 401 L=1,3
401 WORK8(L,J)=WORK9(L,1)
    DO 402 L=1,K
402 WORK8(L+3,J)=-QARRA(L,1)
C
C   RELATIVE COORDINATES MODAL MATRIX CALCULATIONS
C
C   IF(P.EQ.2) GO TO 280
C
C   THREE ARM ROTOR CAL'S
C
C   DO 277 L=1,54
C   DO 277 J=1,3
277 WORK9(L+3,J)=QARRA(L,J)
C
C   WORK9(1-57,1-57)=ARRAY QY6
C
C   DO 278 L=1,57
C   DO 278 J=1,57
C   WORK11(L,J)=0.
C   DO 278 K=1,57
278 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK8(K,J)
C
C   WORK11(1-57,1-57)=MODAL MATRIX EIG-YR
C
C   CALL ANORM(WORK6,WORK11,57,72,57)
C   GO TO 285
C
C   TWO ARM ROTOR CAL'S
C
280 DO 281 L=1,36
    DO 281 J=1,3
281 WORK9(L+3,J)=QARRA(L,J)
C
C   WORK9(1-39,1-39)=ARRAY QT6
C
C   DO 282 L=1,39
C   DO 282 J=1,39
C   WORK11(L,J)=0.
C   DO 282 K=1,39

```

IV G LEVEL 1, MOD 1

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282 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*WORK8(K,J)

C
C
C

WORK11(1-39,1-39)=MODAL MATRIX EIG-TR

CALL ANORM(WORK6,WORK11,57,72,39)

285 CONTINUE

DO 297 L=1,3

DO 296 J=1,3

296 ALPH1(L,J)=0.

297 ALPH1(L,L)=1.

C
C
C

ALPH1(1-3,1-3)=ARRAY I3

DO 298 L=1,57

DO 298 J=1,57

298 WORK11(L,J)=0.

CALL STACK(ALPH1,WORK11,3,3,57,1,1)

CALL STACK(ALPH1,WORK11,3,3,57,10,4)

CALL STACK(ALPH1,WORK11,3,3,57,4,7)

CALL STACK(ALPH1,WORK11,3,3,57,13,10)

CALL STACK(ALPH1,WORK11,3,3,57,7,13)

CALL STACK(ALPH1,WORK11,3,3,57,16,16)

C
C
C

WORK11(1-18,1-18)=ARRAY Q8

DO 299 L=1,18

DO 299 J=1,18

299 WORK11(L+21,J+21)=WORK11(L+21,J+21)+WORK11(L,J)

DO 305 L=4,18

DO 305 J=4,18

305 WORK11(L,J)=0.

DO 306 L=1,18

DO 306 J=1,18

306 WORK11(L+3,J+3)=WORK11(L+3,J+3)+WORK11(L+21,J+21)

C
C
C

WORK11(1-39,1-39)=ARRAY Q9T

IF(P.EQ.2) GO TO 310

C
C
C

THREE ARM ROTOR CALCULATIONS

DO 307 L=1,18

DO 307 J=1,18

307 WORK11(L+39,J+39)=WORK11(L+39,J+39)+WORK11(L+21,J+21)

C
C
C

WORK11(1-57,1-57)=ARRAY Q9Y

DO 308 L=1,57

DO 308 J=1,57

WORK9(L,J)=0.

DO 308 K=1,57

308 WORK9(L,J)=WORK9(L,J)+WORK11(L,K)*WORK6(K,J)

C
C

WORK9(1-57,1-57)=ARRAY EIG-YRF

IV G LEVEL 1, MOD 1

RSMOD1

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```

C      GO TO 315
C
C      TWO ARM ROTOR CALCULATIONS
C
310 CONTINUE
    DO 311 L=1,39
    DO 311 J=1,39
    WORK9(L,J)=0.
    DO 311 K=1,39
311 WORK9(L,J)=WORK9(L,J)+WORK11(L,K)*WORK6(K,J)
C
C      WORK9(1-39,1-39)=ABRAY EIG-TRF
C
315 CONTINUE
    ER=57
    IF(P.EQ.2) ER=39
    EC=ER
    DO 316 L=1,ER
    DO 316 J=1,EC
316 WORK11(L,J)=WORK9(L,J)
    CALL ANORM(WORK11,WORK11,57,57,ER)
    IF(P.EQ.2) GO TO 300
    WRITE(IOUT,286)
286 FORMAT('0','THREE ARM ROTOR MODAL MATRIX')
    NB=-9
    NL=0
    NUMM=28
    NUM1=29
    NBB=19
    NLL=28
    NLL1=57
    GO TO 287
300 WRITE(IOUT,301)
301 FORMAT('0','TWO ARM ROTOR MODAL MATRIX')
    NB=-9
    NL=0
    NUMM=19
    NUM1=20
    NBB=10
    NLL=19
    NLL1=39
287 WRITE(IOUT,288)
288 FORMAT('0','SYMMETRIC MODAL MATRIX EIGENVALUES')
289 NL=NL+10
    IF(NL.GT.NUMM) NL=NUMM
    NB=NB+10
    WRITE(IOUT,290) (I,I=NB,NL)
290 FORMAT('0',7X,'OMEGA1'/'/' EIG. VAL-NO.',I3,9I12)
    WRITE(IOUT,291) (OMEGA1(K),K=NB,NL)
291 FORMAT(12X,10E12.5)
    IF(NL.LT.NUMM) GO TO 289
    WRITE(IOUT,292)
292 FORMAT('0','ANTISYMMETRIC MODAL MATRIX EIGENVALUES')

```

IV G LEVEL 1, MOD 1

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```
NL=0
NB=-9
293 NL=NL+10
    NLL=NLL+10
    IF(NL.GT.NUM1) NL=NUM1
    IF(NLL.GT.NLL1) NLL=NLL1
    NB=NB+10
    NBB=NBB+10
    WRITE(IOUT,294) (I,I=NBB,NLL)
294 FORMAT('0',7X,'OMEGA2'//' EIG. VAL-NO.',I3,9I12)
    WRITE(IOUT,295) (OMEGA2(K),K=NB,NL)
295 FORMAT(12X,10E12.5)
    IF(NL.LT.NUM1) GO TO 293
    WRITE(IOUT,350)
350 FORMAT('0','ROTOR MODAL MATRIX')
    CALL WILLY('EIG-TYRP',WORK11,57,57,1,ER,1,EC)
C
C   INSERT PUNCH-TAPE ROUTINE HERE
    CALL PWRIT(WORK11,57,57,1,ER,1,EC,1,TREIG,IRIT)
C
    CALL RDAMP1
    RETURN
    END
```

IV G LEVEL 1. MOD 1.

RDAMP

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```

SUBROUTINE RDAMP
COMMON /PUNCH/ IRIT,TREIG,TFHFF,TSMAT,ICTYA,TK7H,ICH,TK6HG,TCHG
COMMON /PRINT/ IOUT,IOIN
COMMON /RDAMPC/

```

```

*DAMPR

```

```

COMMON /RHMST/

```

```

*WORK6(72,72),WORK8(72,60),WCRK9(72,60),WORK11(57,57)

```

```

COMMON /RHMST1/

```

```

*XRN(3),XRD1(3),XRD2(3),ZV(3),ALPH1(3,3),GXN(3,3),GZP(3,3),ANG(2),

```

```

*PT1,PT2,PT3,P,BKS1,TEST,QARRA(54,3),XRD3(3)

```

```

COMMON /EIGVEC/

```

```

*COLK(435),COLM(435),EIG(29,29),QMEGA1(28),QMEGA2(29),TOL1,FREQ

```

```

INTEGER P

```

```

DIMENSION AINV(57,57),LCC(57)

```

```

M=57

```

```

IF(P.EQ.2) M=39

```

```

DO 10 L=1,M

```

```

DO 10 J=1,M

```

```

10 AINV(L,J)=WORK8(L,J)

```

C

C

```

AINV(1-39/57,1-39/57)=ARRAY M2T/M2Y

```

C

```

RETURN

```

C

C

```

ENTRY POINT

```

C

```

ENTRY RDAMP1

```

```

QMEGA2(1)=0.

```

```

LZ=0

```

```

LL=28

```

```

LM=LL+1

```

```

LK=57

```

```

IF(P.EQ.3) GO TO 15

```

```

LL=19

```

```

LM=LL+1

```

```

LK=39

```

```

15 CONTINUE

```

```

DO 20 L=1,M

```

```

DO 20 J=1,M

```

```

20 WORK9(L,J)=0.

```

```

DO 21 L=1,LL

```

```

21 WORK9(L,L)=QMEGA1(L)

```

```

DO 22 L=LM,LK

```

```

LZ=LZ+1

```

```

22 WORK9(L,L)=QMEGA2(LZ)

```

C

C

C

```

WORK9(1-39/57,1-39/57)=ARRAY OMEGA-T/Y

```

```

DO 35 L=1,M

```

```

DO 35 J=1,M

```

```

35 WORK6(L,J)=WORK8(L,J)

```

```

CALL ANORM(WORK6,WORK6,72,72,M)

```

```

DO 23 L=1,M

```

```

DO 23 J=1,M

```

IV G LEVEL 1. MOD 1

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23 WORK8(L,J)=AINV(L,J)

DO 24 L=1,M

DO 24 J=1,M

24 AINV(L,J)=WORK6(L,J)

C

C

C

AINV(1-39/57,1-39/57)=ARRAY EIGVEC(NORMALIZED)T/Y

CALL SREVN1(AINV,M,LOC,57,NNN)

DO 25 L=1,M

DO 25 J=1,M

WORK11(L,J)=0.

DO 25 K=1,M

25 WORK11(L,J)=WORK11(L,J)+WORK9(L,K)*AINV(K,J)

C

C

C

WORK11(1-39/57,1-39/57)=ARRAY(OMEGA-T/Y+EIGVEC-NORM(INVERSE))=A

DO 26 L=1,M

DO 26 J=1,M

WORK9(L,J)=0.

DO 26 K=1,M

26 WORK9(L,J)=WORK9(L,J)+WORK6(L,K)*WORK11(K,J)

C

C

C

WORK9(1-39/57,1-39/57)=ARRAY(EIGVEC(NORM)*A)=B

DAMPR2=2.*DAMPR

DO 27 L=1,M

DO 27 J=1,M

WORK11(L,J)=0.

DO 27 K=1,M

27 WORK11(L,J)=WORK11(L,J)+DAMPR2*WORK8(L,K)*WORK9(K,J)

C

C

C

WORK11(1-39/57,1-39/57)=ARRAY CT/Y

DO 29 L=1,M

DO 28 J=1,M

28 WORK9(L,J)=0.

29 WORK9(L,L)=1.

MM=M-3

DO 30 L=1,MM

DO 30 J=1,3

30 WORK9(L+3,J)=-QARRA(L,J)

C

C

C

WORK9(1-39/57,1-39/57)=ARRAY GT7/CY7

DO 31 L=1,M

DO 31 J=1,M

AINV(L,J)=0.

DO 31 K=1,M

31 AINV(L,J)=AINV(L,J)+WORK11(L,K)*WORK9(K,J)

DO 40 L=1,M

40 AINV(L,1)=0.

C

C

C

AINV(1-39/57,1-39/57)=ARRAY CTA/CYA

I. IV. G. LEVEL 1. MOD 1

RDAMP

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WRITE(IOUT,60)

60 FORMAT('0','ROTOR DAMPING MATRIX')

CALL WILLY('ARY-CTYA',AINV,57,57,1,M,1,M)

C

C INSERT TAPE-PUNCH ROUTINE HERE

CALL PWRT(AINV,57,57,1,M,1,M,4,TCTYA,IRIT)

C

RETURN

END

AN IV G LEVEL 1, MOD 1 HMMOD DATE = 71363 13.21.51

```

SUBROUTINE HMMOD
  INTEGER P,ER,EC
  COMMON /PUNCH/ IRIT,TREIG,TFHF,TSNAT,TCTYA,TK7H,TCH,TK6HG,TCHG
  COMMON /BEAM1/
  *HL(3),AL(3),PSS(3),HUB,EI,AE,GJ
  COMMON /RHMST/
  *WORK6(72,72),WORK8(72,60),WCRK9(72,60),WCRK11(57,57)
  COMMON /EIGVEC/
  *COLK(435),COLM(435),EIG(29,29),OMEGA1(28),OMEGA2(29),TCL1,FREQ
  COMMON /HUBID/
  *CI1(3,3),CI2(3,3),CI3(3,3),YHE(3),CM1,CM2,CM3,YHA(3),YFG(3),
  *YHC1(3),YHC2(3),YHC3(3)
  COMMON /PRINT/
  *IOUT,IOIN
  COMMON /TERUP/BRAK
  DIMENSION YHE3(3),GYHL(3,3)
  DO 10 L=1,24
  DO 9 J=1,24
  9 WORK6(L,J)=0.
  10 WORK6(L,L)=1.
  YHE3(3)=0.
  YHE3(2)=0.
  YHE3(1)=-(YHE(1)-YHC3(1))
  CALL GAMMA(GYHL,YHE3)
  CALL STACK(GYHL,WORK6,3,3,72,19,22)
C
C   WORK6(1-24,1-24)=ARRAY Q1H
C
  DO 12 L=1,24
  DO 12 J=1,24
  WORK6(L,J+48)=0.
  DO 12 K=1,24
  12 WORK6(L,J+48)=WORK6(L,J+48)+WCRK6(L,K+24)*WORK6(K,J)
C
C   WORK6(1-24,49-72)=ARRAY(K4H*G1H)
C
  DO 13 L=1,24
  DO 13 J=1,24
  13 WORK6(L,J+24)=WORK6(J,L)
C
C   WORK6(1-24,25-48)=ARRAY Q1H(TRANSPCSE)
C
  DO 14 L=1,24
  DO 14 J=1,24
  WORK6(L,J)=0.
  DO 14 K=1,24
  14 WORK6(L,J)=WORK6(L,J)+WCRK6(L,K+24)*WORK6(K,J+48)
C
C   WORK6(1-24,1-24)=ARRAY K5H
C
  DO 15 L=1,24
  DO 15 J=1,18
  15 WORK8(L,J)=WORK6(L,J+6)
C

```

```

AN IV G LEVEL 1, MOD 1          HMMOD          DATE = 71363          13.21.59

C      WORK8(1-24,1-18)=ARRAY K6H
C      WORK8(1-6,1-18)=ARRAY K6HG
C
      DO 16 L=1,72
      DO 16 J=1,72
16     WORK6(L,J)=0.
      DO 17 L=1,18
      DO 17 J=1,18
17     WORK6(L,J)=WORK8(L+6,J)
C
C      WORK6(1-18,1-18)=ARRAY K7H
C
      DO 18 L=1,3
      WORK6(L,L+39)=CM1
      WORK6(L+6,L+45)=CM2
18     WORK6(L+12,L+51)=CM3
      CALL STACK(CI1,WORK6,3,3,72,4,43)
      CALL STACK(CI2,WORK6,3,3,72,10,49)
      CALL STACK(CI3,WORK6,3,3,72,16,55)
C
C      WORK6(1-18,40-57)=ARRAY MH
C
      CALL COMPR(WORK6,COLK,72,1,1,18)
      CALL COMPR(WORK6,COLM,72,1,40,18)
      CALL LATE(18,COLK,18,TCL1,OMEGA1,NIX3,CCLM,WORK6,72,1,OMEGA2,EIG)
      IF(BRAK.EQ.1.) RETURN
C
C      WORK6(1-18,1-18)=ARRAY EIGVEC-HUB
C
      CALL ANORM(WORK11,WORK6,72,57,18)
C
C      WORK11(1-18,1-18)=NORMALIZE ARRAY EIGVEC-HUB
C      HUB DAMPING MATRIX CALCULATIONS
C
      CALL HDAMP
C
C      INSERT-TAPE-PUNCH ROUTINE HERE
      CALL PWRIT(WORK8,72,60,1,6,1,18,7,TK6HG,IRIT)
      CALL PWRIT(WORK8,72,60,7,24,1,18,5,TK7H,IRIT)
C
      DO 19 L=1,18
      DO 19 J=1,18
19     WORK6(L,J)=0.
      DO 20 L=1,3
      WORK6(L,L)=1.
      WORK6(L+9,L+3)=1.
      WORK6(L+3,L+6)=1.
      WORK6(L+12,L+9)=1.
      WORK6(L+6,L+12)=1.
20     WORK6(L+15,L+15)=1.
C
C      WORK6(1-18,1-18)=ARRAY Q8
C
      DO 21 L=1,18

```

AN IV G LEVEL 1, MOD 1

HMMOD

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```

DO 21 J=1,18
  WORK6(L,J+18)=0.
DO 21 K=1,18
21 WORK6(L,J+18)=WORK6(L,J+18)+WORK6(L,K)*WORK11(K,J)
DO 22 L=1,18
DO 22 J=1,18
22 WORK11(L,J)=WORK6(L,J+18)
  CALL ANORM(WORK11,WORK11,57,57,18)
C
C   WORK11(1-18,1-18)=ARRAY FH-HF
C
  ER=18
  EC=18
  NB=-9
  NL=0
  NUMM=18
  WRITE(IOUT,24)
24 FORMAT('0','HUB-MODAL MATRIX EIGENVALUES')
26 NL=NL+10
  IF(NL.GT.NUMM) NL=NUMM
  NB=NB+10
  WRITE(IOUT,28) (I,I=NB,NL)
28 FORMAT('0',7X,'OMEGA'// ' EIG. VAL-NO.',I3,9I12)
  WRITE(IOUT,30) (OMEGA1(K),K=NB,NL)
30 FORMAT(12X,10E12.5)
  IF(NL.LT.NUMM) GO TO 26
  WRITE(IOUT,40)
40 FORMAT('0','HUB MODAL MATRIX')
  CALL WILLY('MODE-MAT',WORK11,57,57,1,ER,1,EC)
C
C   INSERT TAPE-PUNCH ROUTINE HERE
  CALL PWRIT(WORK11,57,57,1,18,1,18,2,TPHHE,IRIT)
C
  RETURN
  END

```


FILED HDAMP FORTRAN P1

CALldata TIME - SHA

```

SUBROUTINE HDAMP
COMMON /PUNCH/ IRIT, TREIG, TPHHF, TSMAT, TCTYA, TK7H, TCH, TK6HG, TCHG
COMMON /PRINT/ IOOUT, IOIN
COMMON /HDAMP C/
*DAMPH
COMMON /RHMST/
*WORK6(72,72), WORK8(72,60), WORK9(72,60), WORK11(57,57)
COMMON /EIGVEC/
*COLK(435), COLM(435), EIG(29,29), OMEGA1(28), OMEGA2(29), TOL1, FREQ
DIMENSION AINV(18,18), LOC(18)
DO 51 L=1,18
DO 50 J=1,18
50 WORK11(L+19,J+19)=0.
51 WORK11(L+19,L+19)=OMEGA1(L)
C
C WORK11(20-37,20-37)=ARRAY OMEGA-H (WH)
C
DO 52 L=1,18
DO 52 J=1,18
52 AINV(L,J)=WORK11(L,J)
C
C AINV(1-18,1-18)=ARRAY EIGVEC(NORMALIZED)
C
CALL SREVN1(AINV,18,LOC,18,NNX)
DO 53 L=1,18
DO 53 J=1,18
WORK11(L+19,J)=0.
DO 53 K=1,18
53 WORK11(L+19,J)=WORK11(L+19,J)+WORK11(L+19,K+19)*AINV(K,J)
C
C WORK11(20-37,1-18)=ARRAY{WH*OH*(INVERSE)}=A
C
DO 54 L=1,18
DO 54 J=1,18
WORK11(L+39,J)=0.
DO 54 K=1,18
54 WORK11(L+39,J)=WORK11(L+39,J)+WORK11(L,K)*WORK11(K+19,J)
C
C WORK11(40-57,1-18)=ARRAY(EIGVEC(NORM)*A)=B
C
DAMP2=2.*DAMPH
DO 55 L=1,18
DO 55 J=1,18
WORK11(L+19,J+39)=0.
DO 55 K=1,18
55 WORK11(L+19,J+39)=WORK11(L+19,J+39)+DAMP2*WORK6(L,K+39)*WORK11(K+39,
*9,J)
C
C WORK11(20-37,40-57)=ARRAY CH
C
DO 60 L=1,18
DO 60 J=1,18
50 AINV(L,J)=WORK8(L+6,J)
C
C AINV(1-18,1-18)=ARRAY K7H

```

FILED HDAMP FORTRAN P1

CALldata TIME - SHA

```

C
CALL SREVN1(AINV,18,LOC,18,NX1)
DO 61 L=1,18
DO 61 J=1,18
WORK8(L,J+19)=0.
DO 61 K=1,18
61 WORK8(L,J+19)=WORK8(L,J+19)+AINV(L,K)*WCRK11(K+19,J+39)
C
C
WORK8(1-18,20-37)=ARRAY(K7H(INV)*CH)
C
DO 62 L=1,6
DO 62 J=1,18
WORK8(L,J+39)=0.
DO 62 K=1,18
62 WORK8(L,J+39)=WORK8(L,J+39)+WORK8(L,K)*WORK8(K,J+19)
C
C
WORK8(1-6,40-57)=ARRAY CHG
C
WRITE(IOUT,80)
80 FORMAT('0','HUB STIFFNESS MATRIX')
CALL WILLY('ARRY-K7H',WORK8,72,60,7,24,1,18)
WRITE(IOUT,81)
81 FORMAT('0','HUB DAMPING MATRIX')
CALL WILLY('ARRY-CH',WORK11,57,57,20,37,40,57)
WRITE(IOUT,82)
82 FORMAT('0','HUB (ATTACHMENT POINT) STIFFNESS MATRIX')
CALL WILLY('ARY-K6HG',WORK8,72,60,1,6,1,18)
WRITE(IOUT,83)
83 FORMAT('0','HUB (ATTACHMENT POINT) DAMPING MATRIX')
CALL WILLY('ARRY-CHG',WORK8,72,60,1,6,40,57)
C
C
INSERT-TAPE-PUNCH ROUTINE HERE
CALL PWRIT(WORK11,57,57,20,37,40,57,6,TCH,IRIT)
CALL PWRIT(WORK8,72,60,1,6,40,57,8,TCHG,IRIT)
C
RETURN
END

```

FILE: LATE FORTRAN P1

CALLDATA TIME - SHA

SUBROUTINE LATE(N,B,NUMEIG,TOL,OMEGA,NIX,A,ARRAY,NA,IR,COR,EIG) I

COMMON/PRINT/IOUT,IOIN I

COMMON /TERUP/BRAK I

SOLVE THE EIGENVALUE PROBLEM I

(A) (X) = (LAMBDA) (B) (X) I

WHERE- I

A = THE NXN MASS MATRIX I

B = THE NXN STIFFNESS MATRIX I

Q = THE EIGENVALUES I

OMEGA = THE FREQUENCIES I

EIG = THE EIGENVECTORS I

NUMEIG= NUMBER OF EIGENVALUES I

N = ORDER OF A,B I

TOL = TOLERANCE LIMIT 0.LE.TOL.LT.1E-9 I

NIX = B ARRAY POSITIVE-DEFINITE TEST I

=0 ARRAY OK I

=+ OVERFLOW I

=- NOT POSITIVE DEFINITE I

ARRAY - EIGENVECTOR ARRAY OUTPUT I

NA - ORDER OF ARRAY I

IR - STARTING ROW OF EIGENVECTOR ARRAY I

COR - FREQ. SHIFT FACTOR ANTISYMMETRIC PROB. ONLY I

DIMENSION A(1),B(1),EIG(N,1),Q(210) I

DIMENSION ARRAY(NA,NA),OMEGA(NUMEIG) I

EQUIVALENCE (DOUBLE,Q(1)) I

DOUBLE PRECISION DOUBLE I

COMMON/WINTER/INDICS I

COMMON/BOND/M,L I

INDICS=-1 I

M=N I

L=1 I

INDEX=(N*N+N)/2+1 I

INDEX1=INDEX-1 I

6 NIX=0 I

FORM CHOLESKY DECOMPOSITION OF B MATRIX I

CALL FUTILE(B,N,NIX) I

IF(NIX)2,7,7 I

2 WRITE(IOUT,41) I

41 FORMAT('0',10X,'STIFFNESS ARRAY NOT POSITIVE DEFINITE',//,11X,'EIG' I

*ENVALUES-MODAL VECTORS NOT COMPUTED',//,11X,'CHECK INPUT DATA') I

BRAK=1. I

RETURN I

FORM A-MATRIX COLUMN INVERSES - STORE IN Q L

7 IK=0 L

DO 10 I=1,N L

DO 11 I1=1,I L

IL=IK+I1 L

11 Q(I1)=B(IL) L

IK=IK+I L

X=-Q(I) L

N IV G LEVEL 1, MOD 1

FUTILE

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```
SUBROUTINE FUTILE(A,N,NIX)
DIMENSION A(1)
DOUBLE PRECISION SUM
EQUIVALENCE (SUM,SUM)
K1 = 1
KK = 0
DO 210 K = 1,N
  KK = KK + K
  IK = KK
  KK1 = KK - 1
  IP (KK1) 60,50,60
50 ASSIGN 100 TO LEAP
  GO TO 70
60 ASSIGN 80 TO LEAP
70 I1 = K1
  DO 140 I = K,N
    SUM = -A(IK)
    GO TO LEAP, (80,100)
80 IJ = I1
  DO 90 KJ = K1,KK1
    SUM = SUM + A(IJ)*A(KJ)
90 IJ = IJ + 1
100 I1 = I1 + I
  IP (I - K) 120,105,120
105 DENOM = -SUM
  IP (DENOM) 980,980,110
110 DENOM = -SQRT(DENOM)
  A(IK) = -DENOM
  GO TO 130
120 A(IK) = SUM / DENOM
130 IK = IK + I
140 CONTINUE
  K1 = K1 + K
210 CONTINUE
  NIX = 0
220 RETURN
980 NIX = -K
  GO TO 220
END
```

FILE: DAGGER FORTRAN P1

CALLDATA TIME - S

```

SUBROUTINE DAGGER(A,M,P,K0,Q)
DIMENSION A(1),P(1),Q(1)
DOUBLE PRECISION SUM
EQUIVALENCE (SUN,SUM)
K = K0
K1 = K + 1
L=1
LL = 0
INDEX = 1
DO 130 I = 2,K1
LJ = INDEX
SUM = 0.
DO 90 J = 1,L
SUM = SUM + A(LJ)*P(J)
90 LJ = LJ + 1
IF (K - L) 100,120,100
100 LL = LL + L
LJ = LL + L
DO 110 J = 1,K
SUM = SUM + A(LJ)*P(J)
110 LJ = LJ + J
120 Q(I-1) = SUN
INDEX = INDEX + L
130 L = I
SUM = 0.
DO 140 I = 1,K
SUM = SUM + P(I)*Q(I)
A(LL+1) = Q(I)
140 LL = LL + 1
A(LL) = SUN
IF (M - K) 150, 200, 150
150 DO 190 L = K1,M
JL = LL+L-K
SUM = 0.
DO 180 J = 1,K
SUM = SUM + P(J)*A(JL)
180 JL=JL+1
LL = LL + L - 1
190 A(LL) = SUM
200 CONTINUE
RETURN
END

```

IV G LEVEL 1, MOD 1

SWITCH

DATE = 71363

13.17.36

```
SUBROUTINE SWITCH(A,M)
DIMENSION A(1)
N = IABS(M)
IP (N - 2) 190,190,90
90 L = (N*(N+1)) / 2
KEY = 1
LOCK = N/2 + 1
IP (M) 100,190,160
100 IP (N - 3) 110,140,110
110 KKT = 3
NKP = N - 1
IMAGE = L
INTO = L - 3
I = 3
DO 130 K = 2, LOCK
DO 120 IK = KKT, NKP
X = A(IK)
A(IK) = A(INTO)
A(INTO) = X
INTO = INTO - I
120 I = I + 1
KKT = NKP + K
NKP = NKP + N - K
IMAGE = IMAGE - K
INTO = IMAGE
130 I = K
140 IF (KEY) 150,190,150
150 KEY = 0
160 LOV2 = L / 2
K = L - 2
DO 170 I = 3, LOV2
X = A(I)
A(I) = A(K)
A(K) = X
170 K = K - 1
IP (KEY) 180,190,180
180 KEY = 0
GO TO 100
190 RETURN
END
```

IV G LEVEL 1, MOD 1

SYMEIG

DATE = 71363

13.16.35

SUBROUTINE SYMEIG(A,M,INDEX,NUMBR,R,TOL)

C MASTER ROUTINE FOR EIGENVALUES AND (OPTIONALLY) EIGENVECTORS OF A
 C REAL SYMMETRIC MATRIX STORED TRIANGULARLY IN CORE. THE HOUSEHOLDER
 C REDUCTION TO TRIDIAGONAL FORM (SUBROUTINE TFORM) IS FOLLOWED BY A
 C BISECTION TECHNIQUE (SUBROUTINE STURM) FOR THE ROOTS AND THEN IN-
 C VERSE ITERATION (SUBROUTINE TRIVEC) FOR THE VECTORS.

C MEANING OF THE PARAMETERS.

C A - THE TRIANGULAR ARRAY CONTAINING THE MATRIX. DURING EXEC-
 C CUTION THE ARRAY IS CHANGED.

C M - ORDER OF A.

C INDEX - INDEX OF FIRST EIGENVALUE REQUIRED.

C NUMBR - NUMBER OF EIGENVALUES REQUIRED.

C R - REAL ARRAY FOR THE ROOTS AND WORKING STORAGE. R MUST CON-
 C TAIN AT LEAST 8*M WORDS AND START ON A D.P. BOUNDARY.

C TOL - TOLERANCE. IF B IS THE MAG. OF LARGEST ROOT, THE ROOTS
 C WILL USUALLY HAVE ERRORS AS LARGE AS B*TOL.

DIMENSION A(1),R(1)

LD = 1 + M

LO = LD + M

LS = LO + M

LP = LS + M

LQ = LP + M

LR = LQ + M

CALL TFORM(A,M,R(LD),R(LC),R(LS),R(LP))

EPS = AMINI(TOL,1.E-3)

CALL STURM(M,INDEX,NUMBR,R(LD),R(LG),R(LS),R(LP),R,EPS)

RETURN

ENTRY SYMVEC(X,LOW,KOUNT,MID)

DIMENSION X(MID,1)

CALL QSVEC(A,R(LD),R(LC),R(LP),R(LQ),R(LR),R(LS),M)

ROOT = R(LS-1)

EPS = ROOT * 3.E-7

K = LOW

DO 200 I = 1,KOUNT

ROOT = AMINI(ROOT-EPS,R(K))

CALL QWIEL(ROOT,X(1,I))

200 K = K + 1

RETURN

END

V. G. LEVEL 1, MOD 1

TRIEQ

DATE = 71363

13.17.45

```
SUBROUTINE TRIEQ(A,Y)
REAL A(1),Y(1)
COMMON /WINTER/INDIC8
COMMON /BOND/ M,L
DOUBLE PRECISION SUM
EQUIVALENCE (SUM,SUM)
L1 = L
LM1 = L1 - 1
MM1 = M - 1
IF (INDIC8) 130,100,100
100 Y(L1) = Y(L1) / A(L1)
IF (MM1) 105,125,105
105 I1 = L1
DO 120 I = L1,MM1
  I1 = I1 + I
  SUM = -Y(I+1)
  IJ = I1
  DO 110 J = L1,I
    SUM = SUM + A(IJ)*Y(J)
110 IJ = IJ + 1
  II = IJ
120 Y(I+1) = -SUM / A(II)
125 IF (INDIC8) 170,140,170
130 II = (M*M + M) / 2 - LM1
140 I = M
145 Y(I) = Y(I) / A(II)
  II = II - I
  I = I - 1
  IF (I - L1) 170,150,150
150 SUN = -Y(I+1)
  IJ = II + L1
  DO 160 J = L1,I
    Y(J) = Y(J) + SUN*A(IJ)
160 IJ = IJ + 1
  GO TO 145
170 RETURN
  END
```

IV G LEVEL 1, MOD 1

TFORM

DATE = 71363

13.16.45

```

SUBROUTINE TFORM(A,N,D,O,S,P)
DIMENSION A(1),D(1),O(1),S(1)
BL = 0.
BU = 0.
OLD = 0.
D(1) = A(1)
K1K1 = 1
N1 = N - 1
DO 230 K = 1,N1
  KP1 = K + 1
  KK = K1K1
  KKP1 = KK + 1
  NK = N - K
  KN = KK + NK
  K1K1 = KN + 1
  SUM = 0.
  DO 100 KJ = KKP1,KN
100  SUM = SUM + A(KJ)*A(KJ)
  S(K) = SUM
  RHO = SQRT(SUM)
  RAD = OLD + RHO
  BL = AMIN1(BL,D(K)-RAD)
  BU = AMAX1(BU,D(K)+RAD)
  IF (K - N1) 120,230,230
120  OLD = RHO
  IF (A(KKP1)) 140,140,130
130  RHO = -RHO
140  O(K) = RHO
  IF (SUM) 150,230,150
150  A(KKP1) = A(KKP1) - RHO
  RHO = 1. / (RHO*A(KKP1))
  A(KK) = RHO
  IJ = KK
  DO 160 J = KP1,N
  IJ = IJ + 1
  O(J) = A(IJ)
160  D(J) = 0.
  II = K1K1
  NI = NK
  DO 190 I = KP1,N
  D(I) = D(I) + A(II)*O(I)
  IJ = II
  II = II + NI
  NI = NI - 1
  IF (NI) 170,190,170
170  X = O(I)
  DO 180 J = I,N1
  IJ = IJ + 1
  D(J+1) = D(J+1) + A(IJ)*X
180  D(I) = D(I) + A(IJ)*O(J+1)
190  D(I) = D(I) * RHO
  SUM = 0.
  DO 200 I = KP1,N
200  SUM = SUM + D(I)*O(I)

```

IV G LEVEL 1, MOD 1

TFORM

DATE = 71363

13.16.45

```
TAU = RHO * SUM * .5
DO 210 I = KP1,N
210 D(I) = D(I) + TAU*O(I)
II = K1K1
NI = NK
DO 220 I = KP1,N
RHO = D(I)
TAU = O(I)
IJ = II
II = II + NI
NI = NI - 1
DO 220 J = I,N
A(IJ) = A(IJ) + RHO*O(J) + TAU*D(J)
220 IJ = IJ + 1
230 D(K+1) = A(K1K1)
O(N1) = A(KKP1)
O(N) = AMIN1(BL,D(N)-RHO)
S(N) = AMAX1(BU,D(N)+RHO)
RETURN
END
```

IV G LEVEL 1, MOD 1

STURM

DATE = 71363

13.16.56

```
SUBROUTINE STURM (N,LIM1,NUMB,D,CFFD,SEC,PFFD,SIGMA,EPS)
DIMENSION D(1),OFFD(1),SEC(1),SIGMA(1),PFFD(1)
DATA HALF / .5/
BL = OFFD(N)
BU = SEC(N)
LIM2 = LIM1 + NUMB - 1
CALL PREP(N,D,SEC,ROOT,LCRD)
N1 = N - 1
IF (N1) 16,200,200
200 TOL = AMAX1(-BL,BU)
OFFD(N) = TOL
TOL = TOL * AMAX1(1.E-15,EPS)
DO 2 I = LIM1,LIM2
SIGMA(I) = BL
2 PFFD(I) = BU
LORD = 0
L = LIM1 - 1
RUTE = 1.E20
GO TO 3
300 DO 400 I = K,L
400 SIGMA(I) = ROOT
3 K = L + 1
IF (K - LIM2) 4,4,16
4 BU = PFFD(K)
ROOT = BU + HALF * (SIGMA(K) - BU)
IF (K - L) 5,7,5
5 DO 6 I = K,LIM2
IF (BU - PFFD(I)) 7,6,7
6 L = I
7 IF (ABS(ROOT - RUTE) - TCL) 300,300,8
8 CALL DET(LORD)
DO 11 I = K,L
IF(I -LORD) 9,9,10
9 SIGMA(I) = ROOT
GO TO 11
10 PFFD(I) = ROOT
11 CONTINUE
RUTE = ROOT
GO TO 4
16 RETURN
END
```

IV. G LEVEL 1, MOD 1

PREP

DATE = 71363

13.17.05

```
SUBROUTINE PREP(N,D,SEC,/RCGT/,/LCRD/)
DIMENSION D(1),SEC(1)
EQUIVALENCE (RD2,RE2),(RD4,RE4)
N1 = N - 1
RETURN
ENTRY DET
RD0 = ROOT
LOW = 0
LAWD = 0
100 RD2 = 0.00
RD4 = 1.00
DO 120 I = LOW,N1
RD4 = D(I+1) - RD0 - RD2
IF (RD4) 120,140,110
110 LAWD = LAWD + 1
120 RE2 = SEC(I+1) / RE4
130 LORD = LAWD
RETURN
140 LAWD = LAWD + 1
IF (RE2) 150,160,150
150 I = I + 1
160 LOW = I + 1
IF (LOW - N1) 100,100,130
END
```

FILE: QSVEC FORTRAN P1

CALLDATA TIME - SH

SUBROUTINE QSVEC(A,D,OPFD,P,Q,R,S,N)
 C SYMMETRIC MATRIX EIGENVECTOR CALCULATION.
 C GIVEN THE ENTRIES (D AND OPFD) OF THE HOUSEHOLDER TRI-DIAGONAL FORM
 C B OF A REAL SYMMETRIC MATRIX A, AND GIVEN A GOOD APPROXIMATE ROOT OF
 C B (AND A) THIS FORTRAN 4 SUBROUTINE COMPUTES A UNIT EIGENVECTOR X
 C OF B, THEN TRANSFORMS IT TO A UNIT VECTOR OF A, USING THE VECTORS
 C STORED IN THE A ARRAY.

DIMENSION A(1),D(1),OPFD(1),P(1),Q(1),R(1),S(1),X(1)
 DOUBLE PRECISION SUM
 COMMON /INFO/ SUM,N,IX,IA

C
 C PART 1. PRELIMINARIES.
 C

IX = 1
 IA = 1
 N1 = N - 1
 N2 = N - 2
 RETURN
 ENTRY QWIEL(ROOT,X)
 ASSIGN 170 TO KOUNT
 TOL = 0.
 DO 100 I = 1,N
 P(I) = D(I) - ROOT
 Q(I) = OPFD(I)
 R(I) = 0.
 TOL = AMAX1(TOL,ABS(D(I)))
 100 X(I) = RDM(X) + .1
 TOL = (TOL + 1.E-15) * 1.E-15

C
 C PART 2. MATRIX DECOMPOSITION.
 C

DO 150 I = 1,N1
 T = ABS(P(I))
 U = ABS(OPFD(I))
 IF (T + U - TOL) 110,120,120
 110 P(I) = TOL
 T = P(I)
 120 IF (T - U) 130,140,140
 130 S(I) = P(I)/OPFD(I)
 IF(ABS(S(I)).LT.TOL) S(I)=TOL
 S(I) = OR(S(I), 1)
 TEMP = Q(I)
 P(I) = OPFD(I)
 Q(I) = P(I+1)
 R(I) = Q(I+1)
 P(I+1) = TEMP - S(I)*Q(I)
 Q(I+1) = -S(I)*R(I)
 GO TO 150
 140 S(I) = OPFD(I)/P(I)
 S(I) = AND(S(I),-2)
 P(I+1) = P(I+1) - S(I)*Q(I)
 150 CONTINUE
 IF (ABS(P(N)) .LT. TOL) P(N) = TOL
 GO TO 210

C

FILE: QSVEC FORTRAN P1

CALLDATA TIME - SH

C PART 3. RIGHT SIDE MODIFICATION.

C

```

170 ASSIGN 330 TO KOUNT
    DO 200 I = 1, N1
      TEMP = AND(S(I), 1)
      IF (TEMP) 180, 190, 180
180 T = X(I)
      X(I) = X(I+1)
      X(I+1) = T - S(I)*X(I)
      GO TO 200
190 X(I+1) = X(I+1) - S(I)*X(I)
200 CONTINUE

```

C

C PART 4. TRIANGULAR SYSTEM SOLUTION.

C

```

210 X(N) = X(N)/P(N)
      X(N1) = (X(N1) - Q(N1)*X(N)) / P(N1)
      DO 220 I = 2, N1
        K = N - I
220 X(K) = (X(K) - Q(K)*X(K+1) - R(K)*X(K+2)) / P(K)

```

C

C PART 5. SCALING TO UNIT VECTOR.

C

```

230 SUM = 0.D0
      M = N
      SCALAR = SQRT(DOTPRO(X, X))
      DO 250 I = 1, N
250 X(I) = X(I)/SCALAR
      GO TO KOUNT, (170, 330, 370)

```

C

C PART 6. TRANSFORMATION BY ORTHOGONAL MATRICES.

C

```

330 L = (N*(N+1))/2 - 4
      DO 360 I = 1, N2
        NI = N - I
        SUM = 0.D0
        M = I + 1
        SCALAR = A(L-1) * DOTPRO(X(NI), A(L))
        IJ = L
        DO 350 J = NI, N
          X(J) = X(J) + SCALAR*A(IJ)
350 IJ = IJ + 1
360 L = L - I - 3
      ASSIGN 370 TO KOUNT
      GO TO 230
370 RETURN
      END

```

FILED AND FORTRAN P1

CALldata TIME -

```
FUNCTION AND(X,Y)
LOGICAL UND,JA,NEIN
REAL NO
EQUIVALENCE(UND,E),(JA,SI),(NEIN,NO)
IF_AG=0
GO TO 1
ENTRY OR(X,Y)
IFLAG=1
1 SI=X
  ND=Y
  UND=JA.AND.NEIN
  IF(IFLAG.EQ.1)UND=JA.OR.NEIN
  AND=E
  IF(IFLAG.EQ.1)OR=E
RETURN
END
```



```
FUNCTION DOTPRO(X,Y)
DIMENSION X(1),Y(1)
DOUBLE PRECISION S,DOTPRO
COMMON /INFO/ S,N,IX,IY
IF (N) 120,120,100
100 JX = 1
    JY = 1
    DO 110 J = 1,N
        S = S + X(JX)*Y(JY)
        JX = JX + IX
110 JY = JY + IY
120 DOTPRO = S
    RETURN
END
```

```
FUNCTION RDM(IX)
DATA IY/5757403/
IY = IY*65539
IF (IY.GE.0) GO TO 6
IY = IY + 2147483647 + 1
6 YFL = IY
RDM = YFL*.4656613E-9
RETURN
ENTRY RDMIN(IX)
IY = IX
RDMIN = IX
RETURN
ENTRY RDMOUT(IX)
IX = IY
RDMOUT = IX
RETURN
END
```

AN IV G LEVEL 1, MOD 1

COMPR

DATE = 71363

13.20.18

```
C      SUBROUTINE COMPR(A,AC,N,IR,IC,K)
C
C      PURPOSE
C      ROUTINE RESTRUCTURES LOWER TRIANGLE K-ORDER ARRAY TO COMPRESSED
C      STORAGE. K-ORDER ARRAY STORED AS SUB-ARRAY IN A
C
C      A-INPUT ARRAY
C      N-ORDER OF A
C      AC-OUTPUT COLUMN VECTOR
C      IR-INITIAL ROW ELEMENT LOCATION-ARRAY A
C      IC-INITIAL COLUMN ELEMENT LOCATION-ARRAY A
C      K-ORDER OF SUBARRAY STORED IN A
C
C      USAGE
C      CALL SUBROUTINE COMPR(A,AC,N,IR,IC,K)
C
C      SUBROUTINE COMPR(A,AC,N,IR,IC,K)
C      DIMENSION A(N,N),AC(435)
C      IK=0
C      DO 10 I=1,K
C      II=IR+I-1
C      DO 10 J=1,I
C      IK=IK+1
C      JJ=IC+J-1
10 AC(IK)=A(II,JJ)
C      RETURN
C      END
```

AN IV G LEVEL 1, MOD 1

ABCP

DATE = 71363 13.20.25

SUBROUTINE ABCP(A,ANG)

```
C
C THIS ROUTINE GENERATES 3X3 MATRIX A
C ANG - INPUT ANGLE
C
C A - 3X3 REAL MATRIX A
C
  DIMENSION A(9)
  IF(ANG.EQ.10.) GO TO 10
  DO 1 I=2,4
1  A(I)=0.
  C1=-.5
  S1=.866025403784
  IF(ANG.EQ.24.) S1=-S1
  A(7)=0.
  A(1)=1.
  A(5)=C1
  A(6)=-S1
  A(8)=S1
  A(9)=C1
  GO TO 12
10 DO 11 I=1,9
11 A(I)=0.
  A(1)=1.
  A(5)=-1.
  A(9)=-1.
12 RETURN
  END
```

IV G LEVEL 1, MOD 1

ANORM

DATE = 71363

13.21.14

```
SUBROUTINE ANORM(AOUT,AIN,NA,NB,N)
DIMENSION AOUT(NB,NB),AIN(NA,NA)
DO 10 L=1,N
AMX=0.
DO 5 J=1,N
5 AMX=AMAX1(ABS(AIN(J,L)),AMX)
IF(AMX.EQ.0.) AMX=1.
DO 10 J=1,N
10 AOUT(J,L)=AIN(J,L)/AMX
RETURN
END
```

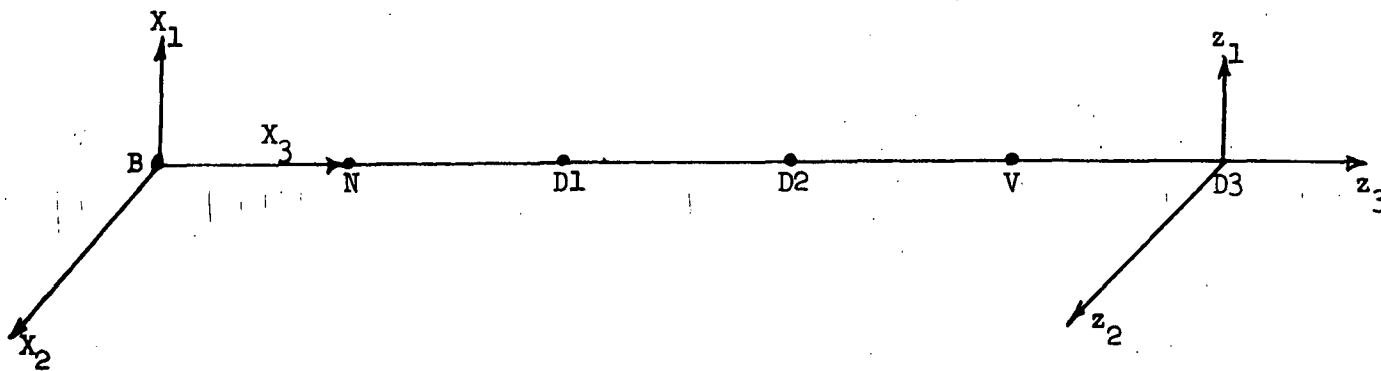
N IV G LEVEL 1, MOD 1

PWRIT

DATE = 71363

13.21.46

```
SUBROUTINE PWRIT(P, NR, NC, SR, ER, SC, EC, IV, TOL, IRIT)
REAL P(NR, NC)
DIMENSION KC(5), KA(5)
INTEGER SR, ER, SC, EC
IF(IRIT.EQ.0) RETURN
J=SC
MOD=1
5 N=0
DO 30 L=SR, ER
IF (ABS(P(L, J)) .LT. TOL) GO TO 16
N=N+1
KC(N)=L
KA(N)=L-(SR-1)
IF(N.EQ.5) GO TO 18
16 IF(L.EQ.ER.AND.N.NE.0) GO TO 18
GO TO 30
18 CONTINUE
WRITE(IRIT, 25) MOD, IV, (KA(I), P(KC(I), J), I=1, N)
25 FORMAT(I3, T79, I2, T4, 5(I3, E12.5))
N=0
30 CONTINUE
J=J+1
MOD=MOD+1
IF(J.LE.EC) GO TO 5
RETURN
END
```



X_1, X_2, X_3 - Fixed Frame in rotor ring. X_3 COORDINATE is aligned with the #1 ARM OF (T or Y) ROTOR CONFIGURATION

z_1, z_2, z_3 - Fixed FRAME IN END module

GEOMETRICAL DATA MEASURED WITH RESPECT
TO X_1, X_2, X_3 COORDINATES

Location Point	Definition	ID
N	Attachment point of rotor arm to rotor ring (X_3 coord in feet)	60
D1	Mass center location of first mass module in rotor arm (X_3 coord in feet)	4
D2	Mass center location of second mass module in rotor (X_3 coord in feet)	7
V	Connection point between rotor arm and associated end module (X_3 coord in feet)	13
D3	Mass center location of end module (X_3 coord in feet)	10
B	Fixed point in hub about which rotor rotates	Not utilized in program

FIGURE 10.53 ROTOR ARM GEOMETRICAL DATA INPUT PROPERTIES

TABLE 10.6
ROTOR INPUT DATA GUIDE

POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
D1	X	2	FT	First mass center location (see Figure 10.53)
	Y	3		
	Z	4		
D2	X	5	FT	Second mass center location (see Figure 10.53)
	Y	6		
	Z	7		
D3	X	8	FT	End module location (see Figure 10.53)
	Y	9		
	Z	10		
V	X	11	FT	Rotor Arm - Associated end module Intersection - Location (see Figure 10.53)
	Y	12		
	Z	13		
—	—	14	—	Number of arms on rotor 2. - T Configuration 3. - Y Configuration
D1	XX	15	Slug-Ft ²	(I) Inertia property data - first mass center location (Coordinate frame nominally aligned to X1, X2, X3 (rotor ring) coordinates - Figure 10.53)
	YX	16		
	ZX	17		
	XY	18		
	YY	19		
	ZY	20		
	XZ	21		
	YZ	22		
	ZZ	23		
D2	XX	24	Slug-Ft ²	(I) Inertia property data - second mass center location (Coordinate frame nominally aligned to X1, X2, X3 (rotor ring) coordinates - Figure 10.53)
	YX	25		
	ZX	26		
	XY	27		
	YY	28		
	ZY	29		

POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
D2	XZ	30	Slug-Ft ²	(I) Inertia property data - second mass center location (Coordinate frame nominally alligned to X1, X2, X3 (rotor ring) coordinates - Figure 10.53
	YZ	31		
	ZZ	32		
D3	XX	33	Slug-Ft ²	(I) Inertia property data - end module (Coordinate frame nominally aligned to X1, X2, X3 (rotor ring) coordinates - Figure 10.53
	XX	34		
	ZX	35		
	XY	36		
	YY	37		
	ZY	38		
	XZ	39		
	YZ	40		
	ZZ	41		
B	XX	42	Slug-Ft ²	(I) Rotor ring inertia property data
	YX	43		
	ZX	44		
	XY	45		
	YY	46		
	ZY	49		
	XZ	50		
	YZ	51		
	ZZ	52		
--	--	51	In ²	Cross-sectional area of beam (rotor arm and/or hub arm)
--	--	52	In ⁴	Area moment of inertia
--	--	53	Lbs/In ²	Modulus of rigidity (G)
--	--	54	Lbs/In ²	Modulus of elasticity (E)
--	--	55	In ⁴	Polar moment of inertia
--	--	56	CPS	Freq shift Utilized in anti-symmetric Elgen-value calculations - value set to 5.
--	--	57	--	Tolerance factor - Utilized in modal matrix calculations - value set to 0. Limits 0 to 1.E-9

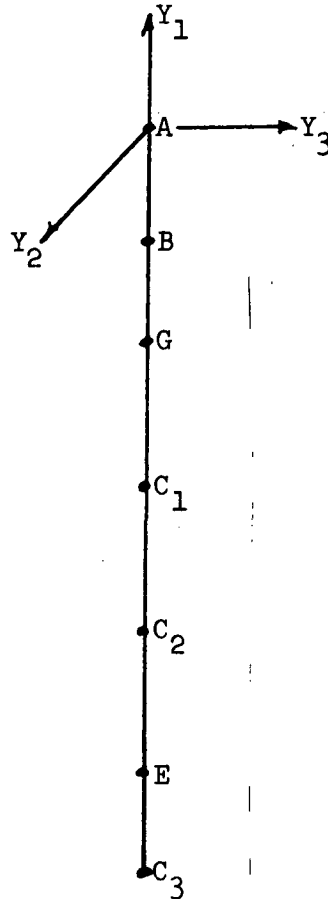
POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
N	X	58	FT	Rotor ring - rotor arm intersection location
	Y	59		
	Z	60		
D1	--	61	Slugs	Point D1 mass
D2	--	62	Slugs	Point D2 mass
D3	--	63	Slugs	End module mass
--	--	64	Rad/Sec	Rotor spin rate
--	--	67	In-Lbs/Rad	Bearing spring constant
--	--	68	--	Structural damping ratio
--	--	100	--	Program control - 0. - Calculate rotor structural data only 1. - Calculate hub data only 2. - Calculate rotor and Hub structural data
		90	--	Program control - punch output data 0. - Do not punch data 7. - Punch data Note: 7. is IBM 360 system output device for punch output
		91	--	Punch output data lower (magnitude) limit; control modal matrix output
--	--	93	--	Punch output data lower (magnitude) limit; control stiffness matrix output
--	--	94	--	Punch output data lower (magnitude) limit; control damping matrix output

TABLE 10.7
SAMPLE ROTOR INPUT DATA SETUP

CARD	FORMATTED DATA INPUT			TYPE
1	1 ROTOR T CONF, RUN NO. 1			Program Control Card
2	4	23.3	XRD1(3)	
3	7	57.3	XRD2(3)	R
4	10	105.23	XRD3(3)	O
5	13	74.25	XRP(3)	T
6	14	2.	P	O
7	15	22731.3	DI1(1,1)	R
8	19	22731.3	DI1(2,2)	
9	23	2083.	DI1(3,3)	D
10	24	22731.3	DI2(1,1)	A
11	28	22731.3	DI2(2,2)	T
12	32	2083.	DI2(3,3)	A
13	33	2034929.53	DI3(1,1)	
14	37	485759.91	DI3(2,2)	I
15	41	1644085.22	DI3(3,3)	N
16	42	11296.	RRIB(1,1)	P
17	46	15586.	RRIB(2,2)	U
18	50	15586.	RRIB(3,3)	T
19	51	15.823	AREA	
20	52	13446.	AI	
21	53	4.0385	E+6GBE	
22	54	1.05	E+7BE	
23	55	26892.	PMJ	
24	56	5.	FREQ	
25	60	6.25	XRN(3)	
26	61	225.16	DML	
27	62	225.16	DM2	
28	63	3881.99	DM3	
29	64	.2618	PR	
30	65	3.14159	ANG(2)*	
31	67	2.1156	E+9BSK1	
32	68	.01	ROTOR DAMPING FACTOR	

*Note: Program data card not required - angle data for rotor arms stored in program

CARD	FORMATTED DATA INPUT	TYPE
33	90 7. IRIT	PUNCH OUTPUT CONTROL
34	91 1. E-5TREIG	MODAL MATRIX TOL. LIMIT
35	93 10. TSMAT	STIFFNESS MATRIX TOL. LIMIT
36	94 10. TCTYA	DAMPING MATRIX TOL. LIMIT
37	BLANK CARD	PROGRAM CONTROL CARD



Y_1, Y_2, Y_3 - Coordinate frame fixed in rigid section of hub at A, the Y_1 axis is parallel to the nominal spin axis of hub.

Geometrical data is measured with respect to Y_1, Y_2, Y_3 coordinates.

Location Point	Definition	Card Data Input Format - I5 I.D. Number
A	Mass center of rigid section of hub	Data not required for program
B	Fixed Point in hub about which rotor rotates	Data not required for program
G	Location of connection between flexible and rigid hub sections	104
C ₁	Location - of first module mass center	107
C ₂	Location - of second module mass center	110
E	Location of connection between hub second module and end module	116
C ₃	Location - of end module	113

FIGURE 10.54 HUB GEOMETRICAL DATA INPUT PROPERTIES

TABLE 10.8

HUB INPUT DATA GUIDE

POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
G	Y ₁	104	Ft	Intersection of hub flexible and rigid sections
	Y ₂	105		
	Y ₃	106		
C ₁	Y ₁	107	Ft	First mass center location
	Y ₂	108		
	Y ₃	109		
C ₂	Y ₁	110	Ft	Second mass center location
	Y ₂	111		
	Y ₃	112		
C ₃	Y ₁	113	Ft	End mass center location
	Y ₂	114		
	Y ₃	115		
E	Y ₁	116	Ft	Hub second module - end module inter-section
	Y ₂	117		
	Y ₃	118		
C ₁	XX	119	Slug-Ft ²	(I) Inertia property data - first mass
	YX	120		
	ZX	121		
	XY	122		
	YY	123		
	ZY	124		
	XZ	125		
	YZ	126		
C ₂	ZZ	127	Slug-Ft ²	(I) Inertia property data - second mass
	XX	128		
	YX	129		
	ZX	130		
	XY	131		
	YY	132		
	ZY	133		
	XZ	134		
YZ	135			
	ZZ	136		

POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
C ₃	XX	137	Slug-Ft ²	(I) Inertia property data - end mass
	YX	138		
	ZX	139		
	XY	140		
	YY	141		
	ZY	142		
	XZ	143		
	YZ	144		
	ZZ	145		
C ₁	--	146	Slugs	Point C1 mass
C ₂	--	147	Slugs	Point C2 mass
C ₃	--	148	Slugs	Point C3 mass
--	--	149	--	Structural damping ratio
--	--	51	In ²	Cross-sectional area of beam (hub arm and/or rotor arm)
--	--	52	In ⁴	Area moment of inertia
--	--	53	Lbs/In ²	Modulus of rigidity (G)
--	--	54	Lbs/In ²	Modulus of elasticity (E)
--	--	55	In ⁴	Polar moment of inertia
--	--	57	--	Tolerance factor-utilized in Eigen value matrix calculations - value set to 0. Limits 0 to 1. E-9
--	--	100	--	Program control 0. - Calculate only rotor structural data 1. - Calculate only hub data 2. - Calculate both rotor and hub
--	--	90	--	Program control - punch output data 0. - Do not punch data 7. - Punch data Note: 7. is IBM 360 system output device for punch output.

POINT LOCATION	COORDINATE REFERENCE	ID NUMBER	DATA UNITS	DEFINITION
--	--	92	--	Punch output data lower magnitude limit; control modal matrix output
--	--	95	--	Punch output data lower magnitude limit; control hub stiffness matrix
--	--	96	--	Punch output data lower magnitude limit; control hub damping matrix
--	--	97	--	Punch output data lower magnitude limit; control hub (attachment point) stiffness matrix
--	--	98	--	Punch output data lower magnitude limit; control hub (attachment point) damping matrix

TABLE 10.9

SAMPLE HUB INPUT DATA SETUP

CARD	FORMATTED DATA INPUT			TYPE
1	LHUB	T CONF. RUN NO. 1		PROGRAM CONTROL CARD
2	100	1.	HUB CALCULATIONS	H
3	104	-27.8	YHG(1)	U
4	107	-63.	YHC1(1)	B
5	110	-122.	YHC2(1)	
6	113	-175.1	YHC3(1)	I
7	116	-156.5	YHE(1)	N
8	119	1735329.45	CI1(1,1)	P
9	123	2012994.74	CI1(2,2)	U
10	127	292228.82	CI1(3,3)	T
11	128	1733430.57	CI2(1,1)	
12	132	1829283.28	CI2(2,2)	D
13	136	98517.36	CI2(3,3)	A
14	137	1644085.2	CI3(1,1)	T
15	141	2034929.53	CI3(2,2)	A
16	145	485759.91	CI3(3,3)	
17	146	741.8022	CM1	
18	147	536.5137	CM2	
19	148	3881.9876	CM3	
20	149	.01	DAMPH	
21	51	15.823	AREA	
22	52	13446.	AI	
23	53	4.0385	E+6GBE	
24	54	1.05	E+7BE	
25	55	26892.	PMJ	
26	90	7.	IRIT	PUNCH OUTPUT CONTROL
27	92	1.	E-5TPHMF	MODAL MATRIX TOL. LIMIT
28	95	10.	TK7H	STIFFNESS MATRIX TOL. LIMIT
29	96	10.	TCH	DAMPING MATRIX TOL. LIMIT
30	97	10.	TK6HG	ATT. PT. STIF. MAT. TOL. LIMIT
31	98	10.	TCHG	ATT. PT. DAMP MAT. TOL. LIMIT
32	BLANK CARD			PROGRAM CONTROL CARD
33	BLANK CARD			PROGRAM CONTROL CARD

10.2.3 Sample Run

Printed output of Rotor and Hub "T" structural data is illustrated in Tables 10.10 and 10.11, respectively. At the start of each printed output the RUN CONTROL CARD along with the BASIC input data are printed.

Rotor Structural Data output will be printed in the following order:

- o Input Card Data
- o Stiffness Matrix
- o Modes (rad/sec)
- o Modal Matrix
- o Damping Matrix

The order of printing hub structural data output is as follows:

- o Input Card Data
- o Hub Stiffness Matrix
- o Hub Damping Matrix
- o Hub (attachment point) Stiffness Matrix
- o Hub (attachment point) damping Matrix
- o Hub Modes (rad/sec)
- o Hub Modal Matrix

The modes are printed in ascending order of frequency (in radians per second) with printed titles designating symmetric and antisymmetric modes. A mode shape, defined by the Modal Matrix columns, is linked with a corresponding frequency whose Eigen value number is identical with the column number.

The modal, stiffness, and damping matrices will be punched on cards in the proper format for direct input to the Time History Program. The last card of each data set of punched output will only contain a zero in column 80. Structural data that is less than the tolerance limit specified by the basic input data will be omitted from the punched output.

TABLE 10.10 SAMPLE OUTPUT: "T." BASELINE ROTOR

RUN NUMBER=	1	ROTOR T CONF.	RUN NO.	1
DD(4) =	23.29999	XRD1(3)		
DD(7) =	57.29999	XRD2(3)		
DD(10) =	105.2300	XRD3(3)		
DD(13) =	74.25000	XRP(3)		
DD(14) =	2.00000	P		
DD(15) =	22731.30	DI1(1,1)		
DD(19) =	22731.30	DI1(2,2)		
DD(23) =	2083.000	DI1(3,3)		
DD(24) =	22731.30	DI2(1,1)		
DD(28) =	22731.30	DI2(2,2)		
DD(32) =	2083.000	DI2(3,3)		
DD(33) =	2034929.	DI3(1,1)		
DD(37) =	485759.9	DI3(2,2)		
DD(41) =	1644085.	DI3(3,3)		
DD(42) =	11296.00	RRIB(1,1)		
DD(46) =	15586.00	RRIB(2,2)		
DD(50) =	15586.00	RRIB(3,3)		
DD(51) =	15.82300	AREA		
DD(52) =	13446.00	AI		
DD(53) =	4038500.	GBE		
DD(54) =	0.1050000E 08	RE		
DD(55) =	26892.00	PMJ		
DD(56) =	5.000000	FREQ		
DD(59) =	6.250000	XRM(3)		
DD(61) =	225.1600	DM1		
DD(62) =	225.1600	DM2		
DD(63) =	3881.990	DM3		
DD(64) =	0.2618000	PR		
DD(65) =	3.141589	ANG(2)		
DD(67) =	0.2115600E 09	BKSI		
DD(68) =	0.9999998E-02	ROTOR DAMPING FACTOR		
DD(90) =	7.000000	IRIT		
DD(91) =	0.1000000E-04	TREIG		
DD(93) =	10.00000	TSMAT		
DD(94) =	10.00000	TCTVA		

TWO ARM ROTOR STIFFNESS MATRIX

TABLE 10.10 (Cont'd)

S-MATRIX

ROW	COLUMN	1	2	3	4	5	6	7	8	9	10
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	-1616.0	0.0	-0.35087E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	-128.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	394.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	-359.57	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	272.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
9	0.0	736.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
10	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
11	-884.27	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
12	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
13	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
14	0.0	-256.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
15	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
16	0.0	28080.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
17	-27998.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
18	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
19	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
20	0.0	-7424.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
21	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
22	0.0	-394.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
23	-359.57	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
24	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
25	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
26	0.0	-272.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
27	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
28	0.0	-736.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
29	-884.27	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
30	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
31	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
32	0.0	256.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
33	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
34	0.0	-28080.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
35	-27998.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
36	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
37	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
38	0.0	7424.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
39	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

S-MATRIX

TABLE 10.10 (Cont'd)

ROW	COLUMN	11	12	13	14	15	16	17	18	19	20
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.50917E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	-0.30036E 06	0.0	-0.50917E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	-0.48865E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.50917E 07	0.0	0.57640E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.57640E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0
9	0.0	0.0	0.0	0.0	-0.22182E 08	0.0	0.0	0.0	0.0	0.0	0.0
10	0.0	0.0	0.0	0.15386E 08	0.0	-0.24180E 07	0.0	0.0	0.0	0.0	0.95386E 08
11	0.27183E 07	0.0	-0.15386E 08	0.0	0.0	0.0	-0.24180E 07	0.0	-0.98019E 07	-0.95386E 08	0.0
12	0.0	0.14683E 08	0.0	0.0	0.0	0.0	0.0	0.0	-0.98019E 07	0.0	0.0
13	-0.15386E 08	0.0	0.34691E 09	0.0	0.0	0.0	0.20478E 08	0.0	0.0	0.75008E 09	0.0
14	0.0	0.0	0.0	0.34691E 09	0.0	-0.20478E 08	0.0	0.0	0.0	0.0	0.75008E 09
15	0.0	0.0	0.0	0.0	0.66677E 08	0.0	0.0	0.0	0.0	0.0	0.0
16	0.0	0.0	0.0	-0.20478E 08	0.0	0.0	0.24180E 07	0.0	0.0	0.0	-0.95386E 08
17	-0.24180E 07	0.0	0.20478E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.95386E 08	0.0
18	0.0	-0.98019E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.98019E 07	0.0	0.0
19	-0.95386E 08	0.0	0.75008E 09	0.0	0.0	0.0	-0.95386E 08	0.0	0.0	0.38218E 10	0.0
20	0.0	0.0	0.0	0.75008E 09	0.0	-0.44495E 08	0.0	0.0	0.0	0.0	0.0
21	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
22	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
23	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
24	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
25	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
26	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
27	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
28	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
29	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
30	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
31	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
32	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
33	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
34	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
35	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
36	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
37	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
38	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
39	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

S-MATRIX

TABLE 10.10 (Cont'd)

ROW	COLUMN	21	22	23	24	25	26	27	28	29	30
1	0.0	0.0	0.0	0.35087E 08	0.0	0.24148E 09	0.0	0.0	0.0	0.0	0.0
2	0.0	0.35087E 08	0.0	0.0	0.0	-0.24148E 09	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.44234E 08	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
9	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
10	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
11	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
12	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
13	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
14	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
15	-0.44495E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
16	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
17	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
18	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
19	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
20	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
21	0.44495E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
22	0.0	0.26761E 07	0.0	0.0	0.0	0.0	-0.15147E 08	0.0	-0.30036E 06	0.0	0.0
23	0.0	0.0	0.26761E 07	0.0	0.0	0.15147E 08	0.0	0.0	0.0	-0.30036E 06	0.0
24	0.0	0.0	0.0	0.14631E 08	0.0	0.0	0.0	0.0	0.0	0.0	-0.48865E 07
25	0.0	0.0	0.15147E 08	0.0	0.34556E 09	0.0	0.0	0.0	0.0	0.50917E 07	0.0
26	0.0	-0.15147E 08	0.0	0.0	0.0	0.0	0.34556E 09	0.0	-0.50917E 07	0.0	0.0
27	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.66416E 08	0.0	0.0	0.0
28	0.0	-0.30036E 06	0.0	0.0	0.0	0.0	-0.50917E 07	0.0	0.27183E 07	0.0	0.0
29	0.0	0.0	-0.30036E 06	0.0	0.0	0.50917E 07	0.0	0.0	0.0	0.27183E 07	0.0
30	0.0	0.0	0.0	0.0	-0.48865E 07	0.0	0.0	0.0	0.0	0.0	0.14688E 08
31	0.0	0.0	-0.50917E 07	0.0	0.0	0.57640E 08	0.0	0.0	0.0	-0.15386E 08	0.0
32	0.0	0.50917E 07	0.0	0.0	0.0	0.0	0.57640E 08	0.0	0.15386E 08	0.0	0.0
33	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.22182E 08	0.0	0.0	0.0
34	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.24180E 07	0.0	0.0
35	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.24180E 07	0.0
36	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.98019E 07
37	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
38	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.95386E 08	0.0	0.0
39	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

S-MATRIX

TABLE 10.10 (Cont'd)

ROW	COLUMN	31	32	33	34	35	36	37	38	39
1	C.0	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0
2	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
3	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
4	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
5	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
6	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
7	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
8	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
9	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
10	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
11	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
12	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
13	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
14	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
15	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
16	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
17	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
18	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
19	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
20	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
21	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
22	C.0	C.50917E 07	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
23	-0.50917E 07	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
24	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
25	0.57640E 08	C.0	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
26	C.0	0.57640E 08	C.0	C.0	0.0	0.0	0.0	0.0	0.0	0.0
27	C.0	C.0	-0.22182E 08	C.0	0.0	0.0	0.0	0.0	0.0	0.0
28	C.0	0.15386E 08	C.0	C.0	-0.24180E 07	0.0	0.0	0.0	0.95386E 08	0.0
29	-0.15386E 08	C.0	C.0	C.0	0.0	-0.24180E 07	0.0	-0.95386E 08	0.0	0.0
30	C.0	C.0	C.0	C.0	0.0	0.0	-0.98019E 07	0.0	0.0	0.0
31	0.34691E 09	C.0	C.0	C.0	0.20478E 08	0.0	0.0	0.75008E 09	0.0	0.0
32	C.0	0.34691E 09	C.0	C.0	-0.20478E 08	0.0	0.0	0.0	0.75008E 09	0.0
33	C.0	C.0	0.56677E 08	C.0	0.0	0.0	0.0	0.0	0.0	-0.44495E 08
34	C.0	-0.20478E 08	C.0	C.0	0.24180E 07	0.0	0.0	0.0	-0.95386E 08	0.0
35	0.20478E 08	C.0	C.0	C.0	0.0	0.0	0.98019E 07	0.0	0.0	0.0
36	C.0	C.0	C.0	C.0	0.0	0.0	0.0	0.38218E 10	0.0	0.0
37	0.75008E 09	C.0	C.0	C.0	0.95386E 08	0.0	0.0	0.0	0.38218E 10	0.0
38	C.0	0.75008E 09	C.0	C.0	-0.95386E 08	0.0	0.0	0.0	0.38218E 10	0.0
39	C.0	C.0	-0.44495E 08	C.0	0.0	0.0	0.0	0.0	0.0	0.44495E 08

TWO ARM ROTOR MODAL MATRIX

TABLE 10.10 (Cont'd)

SYMMETRIC MODAL MATRIX EIGENVALUES

OMEGA1

EIG. VAL-NO. 1 2 3 4 5 6 7 8 9 10
 0.86054E 00 0.92989E 00 0.24699E 01 0.88844E 01 0.14604E 02 0.24638E 02 0.53133E 02 0.56732E 02 0.10384E 03 0.10502E 03

OMEGA1

EIG. VAL-NO. 11 12 13 14 15 16 17 18 19
 0.10625E 03 0.14362E 03 0.14504E 03 0.15093E 03 0.16340E 03 0.16709E 03 0.21162E 03 0.21257E 03 0.29717E 03

ANTISYMMETRIC MODAL MATRIX EIGENVALUES

OMEGA2

EIG. VAL-NO. 20 21 22 23 24 25 26 27 28 29
 0.71603E-01 0.81744E 00 0.25968E 01 0.56783E 01 0.12317E 02 0.24638E 02 0.32570E 02 0.44683E 02 0.76819E 02 0.84573E 02

OMEGA2

EIG. VAL-NO. 30 31 32 33 34 35 36 37 38 39
 0.11201E 03 0.11383E 03 0.14599E 03 0.15087E 03 0.16786E 03 0.20641E 03 0.21133E 03 0.29576E 03 0.31660E 03 0.34149E 03

ROTOR MODAL MATRIX

EIG-TYPE

TABLE 10.10 (Cont'd)

ROW	COLUMN	1	2	3	4	5	6	7	8	9	10
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.48204E-14	0.16047E-12	0.10495	0.36674E-09	0.15040E-08	0.86863E-12	0.21421E-05	0.18308E-05	0.64847E-03	0.99071E-03	
4	0.33215E-07	0.43656E-01	0.89379E-06	0.11042E-04	0.24449	0.14343E-05	0.46491E-02	-1.0000	-0.69522E-01	-1.0000	
5	0.42930E-01	0.47507E-07	0.19144E-05	0.22743	-0.34037E-05	0.15929E-05	1.0000	0.34105E-02	-1.0000	0.68526E-01	
6	0.81526E-08	0.60791E-09	0.26379E-10	0.12922E-05	0.22712E-05	0.25608	0.23604E-03	0.24194E-03	0.16738E-02	0.13165E-02	
7	0.83795E-07	0.34176	0.27605E-08	0.17222E-04	-1.0000	-0.78583E-06	0.14891E-02	0.43786	0.51916E-01	0.81679	
8	0.33855	-0.14593E-06	0.25199E-08	-1.0000	0.18919E-04	0.57529E-07	0.78897	0.38672E-02	0.92102	-0.64116E-01	
9	0.43074E-12	0.23329E-13	0.77523E-12	0.32450E-08	0.15911E-07	0.75958	-0.26367E-06	0.25353E-05	0.10265E-03	0.83804E-04	
10	0.27977E-06	1.0000	-0.14800E-08	0.41345E-06	0.24751E-01	0.47531E-07	0.10883E-03	0.31835E-01	0.10907E-02	0.16297E-01	
11	1.0000	-0.42573E-06	0.19878E-08	0.15615	0.49691E-05	0.18249E-06	0.48274E-01	0.20075E-03	0.23248E-01	0.16121E-02	
12	0.42417E-12	0.25292E-13	0.15038E-11	0.27308E-08	0.17064E-07	-1.0000	-0.44489E-05	0.44835E-05	0.30550E-04	0.24179E-04	
13	-0.44807E-02	0.75674E-08	0.12383E-09	0.23253E-01	0.60768E-06	0.64041E-07	0.87450E-01	0.38612E-03	0.60357E-01	0.46279E-02	
14	-0.15390E-08	0.49550E-02	0.98324E-10	0.33888E-06	0.24736E-01	0.23195E-07	0.24862E-03	0.85726E-01	0.27871E-02	0.61263E-01	
15	0.94818E-14	0.38312E-12	0.25087	0.72924E-09	0.35382E-08	0.19866E-11	0.40579E-05	0.33418E-05	0.31895E-03	0.44435E-03	
16	-0.11929E-01	0.50778E-08	0.25167E-10	0.94213E-02	0.30881E-06	0.59397E-07	0.94594E-01	0.44358E-03	0.73103E-01	0.50693E-02	
17	0.28266E-08	0.11958E-01	0.13287E-10	0.28000E-07	0.63756E-02	0.64555E-07	0.37011E-03	0.10305	-0.60129E-02	0.93207E-01	
18	-0.23378E-10	0.22762E-11	0.75094	0.50531E-08	0.75765E-08	0.64715E-10	0.28205E-06	0.11845E-05	0.17290E-03	0.25711E-03	
19	-0.14173E-01	0.59871E-08	0.10530E-09	0.23915E-01	0.41383E-06	0.12999E-07	0.46326E-02	0.19612E-04	0.19458E-02	0.13508E-03	
20	0.43306E-08	0.14083E-01	0.10514E-09	0.42667E-06	0.26305E-01	0.33776E-07	0.53759E-04	0.14386E-01	0.41135E-03	0.62099E-02	
21	0.18089E-12	0.65077E-12	1.0000	-0.59416E-10	0.15040E-08	0.56009E-12	0.21178E-05	0.18009E-05	0.64938E-03	0.99216E-03	
22	0.33215E-07	0.43656E-01	0.89379E-06	0.11042E-04	0.24449	0.14343E-05	0.46491E-02	-1.0000	-0.69522E-01	-1.0000	
23	-0.42930E-01	0.47507E-07	0.19144E-05	0.22743	0.34037E-05	0.15929E-05	-1.0000	-0.34105E-02	1.0000	-0.68526E-01	
24	0.81526E-08	0.60791E-09	0.26379E-10	0.12922E-05	0.22712E-05	0.25608	0.23604E-03	0.24194E-03	0.16738E-02	0.13165E-02	
25	0.83795E-07	0.34176	0.27605E-08	0.17222E-04	-1.0000	-0.78583E-06	0.14891E-02	0.43786	0.51916E-01	0.81679	
26	-0.33855	0.14593E-06	0.25199E-08	1.0000	-0.18919E-04	0.57529E-07	0.78897	-0.38672E-02	0.92102	0.64116E-01	
27	0.43074E-12	0.23329E-13	0.77523E-12	0.32450E-08	0.15911E-07	0.75958	-0.26367E-06	0.25353E-05	0.10265E-03	0.83804E-04	
28	0.27977E-06	1.0000	-0.14800E-08	0.41345E-06	0.24751E-01	0.47531E-07	0.10883E-03	0.31835E-01	0.10907E-02	0.16297E-01	
29	-1.0000	0.42573E-06	0.19878E-08	0.15615	0.49691E-05	0.18249E-06	0.48274E-01	0.20075E-03	0.23248E-01	0.16121E-02	
30	0.42417E-12	0.25292E-13	0.15038E-11	0.27308E-08	0.17064E-07	-1.0000	-0.44489E-05	0.44835E-05	0.30550E-04	0.24179E-04	
31	0.44807E-02	0.75674E-08	0.12383E-09	0.23253E-01	0.60768E-06	0.64041E-07	0.87450E-01	0.38612E-03	0.60357E-01	0.46279E-02	
32	-0.15390E-08	0.49550E-02	0.98324E-10	0.33888E-06	0.24736E-01	0.23195E-07	0.24862E-03	0.85726E-01	0.27871E-02	0.61263E-01	
33	-0.94818E-14	0.38312E-12	0.25087	0.72924E-09	0.35382E-08	0.19866E-11	0.40579E-05	0.33418E-05	0.31895E-03	0.44435E-03	
34	0.11929E-01	0.50778E-08	0.25167E-10	0.94213E-02	0.30881E-06	0.59397E-07	0.94594E-01	0.44358E-03	0.73103E-01	0.50693E-02	
35	0.28266E-08	0.11958E-01	0.13287E-10	0.28000E-07	0.63756E-02	0.64555E-07	0.37011E-03	0.10305	-0.60129E-02	0.93207E-01	
36	0.23378E-10	0.22762E-11	0.75094	0.50531E-08	0.75765E-08	0.64715E-10	0.28205E-06	0.11845E-05	0.17290E-03	0.25711E-03	
37	0.14173E-01	0.59871E-08	0.10530E-09	0.23915E-01	0.41383E-06	0.12999E-07	0.46326E-02	0.19612E-04	0.19458E-02	0.13508E-03	
38	0.43306E-08	0.14083E-01	0.10514E-09	0.42667E-06	0.26305E-01	0.33776E-07	0.53759E-04	0.14386E-01	0.41135E-03	0.62099E-02	
39	-0.18089E-12	0.65077E-12	-1.0000	0.59416E-10	0.15040E-08	0.56009E-12	0.21178E-05	0.18009E-05	0.64938E-03	0.99216E-03	

EIG-TYPR

TABLE 10.10 (Cont'd)

ROW	COLUMN	11	12	13	14	15	16	17	18	19	20
1	C.0	0.0	0.0	C.0	0.0	0.0	0.0	C.0	0.0	0.0	1.0000
2	0.0	0.0	0.0	0.0	0.0	0.0	C.0	0.0	0.0	0.0	0.0
3	0.99827	-0.1185R-03-C.38076E-03	0.48687E-04	0.27507	0.41099E-04	0.26999E-03	0.17865	0.11799E-05	0.0	0.0	0.0
4	-0.16678E-01	-0.72970	1.0000	-0.14831E-01	0.17737E-02	0.73190E-01	0.13640E-01	0.11411E-01	0.19379E-01	0.0	0.0
5	0.1431E-01	-1.0000	-0.71040	0.13793	-0.15726E-02	0.13014E-02	0.97461E-02	0.8166E-02	0.14530E-01	0.0	0.0
6	0.51857E-04	0.29299E-02	-0.16092E-01	0.97289E-03	0.13597E-03	0.61634E-03	1.0000	0.44348E-03	0.89064	0.0	0.0
7	0.14461E-01	0.59006E-01	0.74284E-01	0.10310E-02	0.53674E-03	1.0000	0.44348E-03	0.29578E-02	0.34693E-03	0.0	0.0
8	-0.14687E-01	0.81021E-01	0.21762E-01	1.0000	-0.15035E-03	0.41933E-03	0.94630	0.66109E-03	0.36683E-03	0.0	0.0
9	0.58336E-05	0.35242E-03	0.26249E-02	0.18397E-03	0.33607E-04	0.17607E-03	0.94630	0.84771	1.0000	0.0	0.0
10	-0.28357E-03	0.77115E-03	0.11917E-02	0.73800E-05	0.33931E-04	0.57480E-01	0.10881E-04	0.16178E-03	0.95921E-05	0.0	0.0
11	0.36629E-03	0.18654E-02	0.28648E-02	0.62031E-01	0.95263E-05	0.12942E-04	0.22618E-04	0.14341E-04	0.55645E-04	0.0	0.0
12	-0.10491E-05	0.57266E-04	0.35896E-03	0.22782E-04	0.35221E-05	0.1734E-04	0.56543E-01	0.50576E-01	0.29431E-01	0.0	0.0
13	-0.11837E-02	0.73898	-0.92981E-01	0.84472E-03	0.14084E-03	0.19754E-04	0.47565E-03	0.21932E-03	0.53175E-03	0.0	0.0
14	-0.12434E-02	0.97248E-01	0.13076	0.18461E-02	0.18372E-03	0.58332E-05	0.12270E-02	0.10218E-02	0.15721E-02	0.0	0.0
15	0.40087	0.13897E-03	0.50069E-03	0.78962E-04	0.63694	-0.10389E-03	0.14857E-02	1.0000	-0.15531E-04	0.0	0.0
16	-0.11376E-02	0.16101E-01	0.12410E-01	0.12135	0.27940E-04	0.84779E-05	0.10161E-03	0.70495E-04	0.14188E-03	0.0	0.0
17	-0.16333E-02	0.90897E-02	0.11933E-01	0.16668E-03	0.56229E-04	0.10774	-0.24494E-04	0.23937E-03	0.47207E-04	0.0	0.0
18	-0.28077	-0.24434E-05	0.54384E-03	0.69204E-04	-1.0000	-0.18779E-03	0.67082E-03	0.48665	0.44511E-04	0.0	0.0
19	0.30486E-04	0.11847E-03	0.19412E-03	0.45233E-02	0.70261E-06	0.79050E-06	0.73142E-06	0.29706E-06	0.16570E-05	0.0	0.0
20	0.10787E-03	0.19581E-03	0.30240E-03	0.17669E-05	0.10488E-04	0.18689E-01	0.35843E-05	0.50107E-04	0.31562E-05	0.0	0.0
21	-1.0000	0.11236E-03	0.38133E-03	0.48648E-04	0.27434	-0.40936E-04	0.27054E-03	0.17905	-0.12241E-05	0.0	0.0
22	-0.16678E-01	0.72970	1.0000	-0.14831E-01	0.17737E-02	0.70190E-01	0.13640E-01	0.11411E-01	0.19379E-01	0.0	0.0
23	-0.1431E-01	1.0000	0.71040	-0.13793	0.15726E-02	0.13014E-02	0.97461E-02	0.61268E-02	0.14530E-01	0.0	0.0
24	0.51857E-04	0.29299E-02	-0.16092E-01	0.97289E-03	0.13597E-03	0.61634E-03	1.0000	0.89064	-0.98472	0.0	0.0
25	0.14461E-01	0.59006E-01	0.74284E-01	0.10310E-02	0.53674E-03	1.0000	0.44348E-03	0.29578E-02	0.34693E-03	0.0	0.0
26	0.14687E-01	0.81021E-01	0.21762E-01	-1.0000	-0.15035E-03	0.41933E-03	0.94630	0.66109E-03	0.36683E-03	0.0	0.0
27	0.58336E-05	0.35242E-03	0.26249E-02	0.18397E-03	0.33607E-04	0.17607E-03	0.94630	0.84771	1.0000	0.0	0.0
28	-0.28357E-03	0.77115E-03	0.11917E-02	0.73800E-05	0.33931E-04	0.57480E-01	0.10881E-04	0.16178E-03	0.95921E-05	0.0	0.0
29	-0.36629E-03	0.18654E-02	0.28648E-02	0.62031E-01	0.95263E-05	0.12942E-04	0.22618E-04	0.14341E-04	0.55645E-04	0.0	0.0
30	-0.10491E-05	0.57266E-04	0.35896E-03	0.22782E-04	0.35221E-05	0.1734E-04	0.56543E-01	0.50576E-01	0.29431E-01	0.0	0.0
31	0.11837E-02	0.73808	0.92981E-01	0.84472E-03	0.14084E-03	0.19754E-04	0.47565E-03	0.21932E-03	0.53175E-03	0.0	0.0
32	-0.12434E-02	0.97248E-01	0.13076	0.18461E-02	0.18372E-03	0.58332E-05	0.12270E-02	0.10218E-02	0.15721E-02	0.0	0.0
33	-0.40087	0.13897E-03	0.50069E-03	0.78962E-04	0.63694	-0.10389E-03	0.14857E-02	1.0000	0.15531E-04	0.0	0.0
34	0.11376E-02	0.16101E-01	0.12410E-01	0.12135	0.27940E-04	0.84779E-05	0.10161E-03	0.70495E-04	0.14188E-03	0.0	0.0
35	-0.16333E-02	0.90897E-02	0.11933E-01	0.16668E-03	0.56229E-04	0.10774	-0.24494E-04	0.23937E-03	0.47207E-04	0.0	0.0
36	0.28077	0.24434E-05	0.54384E-03	0.69204E-04	1.0000	-0.18779E-03	0.67082E-03	0.48665	-0.44511E-04	0.0	0.0
37	-0.30486E-04	0.11847E-03	0.19412E-03	0.45233E-02	0.70261E-06	0.79050E-06	0.73142E-06	0.29706E-06	0.16570E-05	0.0	0.0
38	0.10787E-03	0.19581E-03	0.30240E-03	0.17669E-05	0.10488E-04	0.18689E-01	0.35843E-05	0.50107E-04	0.31562E-05	0.0	0.0
39	1.0000	-0.11236E-03	0.38133E-03	0.48648E-04	0.27434	-0.40936E-04	0.27054E-03	0.17905	0.12241E-05	0.0	0.0

TABLE 10.10 (Cont'd)

ROW	COLUMN	21	22	23	24	25	26	27	28	29	30
1	-0.31729E-07	0.98861E-10	0.97426E-02	0.31229E-07	0.23182E-10	0.94371E-02	0.11722E-07	0.95362E-02	0.11332E-06	0.94702E-02	
2	-0.31742E-02	0.1891E-07	0.29607E-07	0.96003E-02	0.12576E-07	0.24958E-08	0.94552E-02	0.97540E-07	0.95255E-02	0.26182E-06	
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	
4	-0.43701E-01	0.44621E-06	0.28237E-07	0.78610E-01	0.42788E-06	0.28398E-07	0.31833E-02	0.16718E-05	0.11753	0.71956E-05	
5	0.32429E-08	0.97309E-10	0.12584E-01	0.10580E-06	0.29734E-08	0.46232E-01	0.17397E-06	0.14479	0.17434E-05	0.27968	
6	-0.33665E-06	0.60999E-06	0.22360E-07	0.10594E-07	0.25608	0.22367E-08	0.67255E-08	0.22908E-07	0.29501E-07	0.19060E-06	
7	-0.34193	-0.21534E-05	0.10054E-05	0.14308	-0.45558E-06	0.14542E-06	0.52289	-0.63498E-05	0.62485	0.12131E-04	
8	0.37222E-06	0.16617E-08	0.21160	-0.75145E-06	0.11137E-07	0.50423	-0.62332E-06	0.64237	0.77272E-05	0.38802	
9	-0.71733E-07	0.36031E-05	0.82459E-09	0.33054E-09	0.75958	-0.65224E-11	0.41812E-08	0.14230E-08	0.86018E-07	0.25894E-07	
10	-1.0000	-0.29854E-05	0.31712E-05	1.0000	-0.16467E-05	0.26207E-06	1.0000	-0.10239E-04	1.0000	0.27598E-04	
11	0.18666E-05	0.80541E-08	-1.0000	-0.24486E-05	0.24843E-07	1.0000	-0.10741E-05	1.0000	0.12011E-04	1.0000	
12	0.54976E-07	0.55228E-08	0.77067E-09	0.10870E-09	1.0000	-0.48673E-10	0.66187E-10	0.45572E-09	0.25102E-08	0.43502E-08	
13	-0.32479E-09	0.14115E-10	0.18211E-02	0.15251E-07	0.41717E-09	0.64833E-02	0.24638E-07	0.19721E-01	0.23459E-06	0.37859E-01	
14	-0.49594E-02	0.28053E-07	0.99362E-08	0.72637E-02	0.60340E-07	0.24360E-08	0.30701E-02	0.18217E-06	0.19051E-01	0.88742E-06	
15	0.11645E-06	-0.25085	-0.34543E-11	0.16429E-07	0.66899E-07	0.18678E-14	0.23003E-07	0.20982E-12	0.98517E-08	0.34148E-11	
16	-0.22867E-07	0.84218E-10	0.10980E-01	0.20581E-07	0.64634E-09	0.16397E-01	0.12258E-07	0.26458E-03	0.14329E-07	0.21267E-01	
17	-0.11958E-01	0.14949E-06	0.35172E-07	0.92406E-02	0.29707E-07	0.37886E-08	0.21921E-01	0.16191E-07	0.12625E-02	0.57164E-06	
18	-0.29399E-08	0.75092	-0.10381E-10	0.10851E-07	0.12494E-06	0.26959E-14	0.11715E-08	0.73355E-13	0.18626E-08	0.38137E-11	
19	-0.33499E-01	0.62539E-07	0.47180E-07	0.17558E-01	0.39226E-07	0.53041E-09	0.97041E-02	0.12747E-07	0.98454E-02	0.91832E-02	
20	-0.14079E-01	0.62539E-07	0.47180E-07	0.17558E-01	0.39226E-07	0.53041E-09	0.97041E-02	0.12747E-07	0.98454E-02	0.91832E-02	
21	0.58642E-09	-1.0000	-0.13774E-10	0.84279E-10	0.18885E-09	0.89393E-15	0.15105E-10	0.11176E-15	0.37369E-11	0.38495E-14	
22	0.43701E-01	0.44621E-06	0.28237E-07	0.78610E-01	0.42788E-06	0.28398E-07	0.31833E-02	0.16718E-05	0.11753	0.71956E-05	
23	0.22232E-08	0.97309E-10	0.12584E-01	0.10580E-06	0.29734E-08	0.46232E-01	0.17397E-06	0.14479	0.17434E-05	0.27968	
24	0.33666E-06	0.60999E-06	0.22360E-07	0.10594E-07	0.25608	-0.22367E-08	0.67255E-08	0.22908E-07	0.29501E-07	0.19060E-06	
25	0.34193	-0.21534E-05	0.10054E-05	0.14308	-0.45558E-06	0.14542E-06	0.52289	-0.63498E-05	0.62485	0.12131E-04	
26	0.37222E-06	0.16617E-08	0.21160	-0.75145E-06	0.11137E-07	0.50423	-0.62332E-06	0.64237	0.77272E-05	0.38802	
27	0.71733E-07	0.36031E-05	0.82459E-09	0.33054E-09	0.75958	-0.65224E-11	0.41812E-08	0.14230E-08	0.86018E-07	0.25894E-07	
28	1.0000	0.29854E-05	0.31712E-05	-1.0000	0.16467E-05	0.26207E-06	-1.0000	0.10239E-04	-1.0000	-0.27598E-04	
29	0.18666E-05	0.80541E-08	-1.0000	-0.24486E-05	0.24843E-07	1.0000	-0.10741E-05	1.0000	0.12011E-04	1.0000	
30	0.54976E-07	0.55228E-08	0.77067E-09	0.10870E-09	1.0000	-0.48673E-10	0.66187E-10	0.45572E-09	0.25102E-08	0.43502E-08	
31	-0.32479E-09	0.14115E-10	0.18211E-02	0.15251E-07	0.41717E-09	0.64833E-02	0.24638E-07	0.19721E-01	0.23459E-06	0.37859E-01	
32	0.49594E-02	0.28053E-07	0.99362E-08	0.72637E-02	0.60340E-07	0.24360E-08	0.30701E-02	0.18217E-06	0.19051E-01	0.88742E-06	
33	0.11645E-06	-0.25085	-0.34543E-11	0.16429E-07	0.66899E-07	0.18678E-14	0.23003E-07	0.20982E-12	0.98517E-08	0.34148E-11	
34	-0.22867E-07	0.84218E-10	0.10980E-01	0.20581E-07	0.64634E-09	0.16397E-01	0.12258E-07	0.26458E-03	0.14329E-07	0.21267E-01	
35	-0.11958E-01	0.14949E-06	0.35172E-07	0.92406E-02	0.29707E-07	0.37886E-08	0.21921E-01	0.16191E-07	0.12625E-02	0.57164E-06	
36	-0.29399E-08	0.75092	-0.10381E-10	0.10851E-07	0.12494E-06	0.26959E-14	0.11715E-08	0.73355E-13	0.18626E-08	0.38137E-11	
37	-0.33499E-07	0.14221E-09	0.17658E-01	0.39226E-07	0.53041E-09	0.87041E-02	0.12747E-07	0.98454E-02	0.11648E-06	0.91832E-02	
38	0.14079E-01	0.62539E-07	0.47180E-07	0.17558E-01	0.39226E-07	0.53041E-09	0.97041E-02	0.12747E-07	0.98454E-02	0.91832E-02	
39	0.58642E-08	-1.0000	-0.13774E-10	0.84279E-10	0.18885E-09	0.89393E-15	0.15105E-10	0.11176E-15	0.37369E-11	0.38495E-14	

TABLE 10.10 (Cont'd)

ROW	COLUMN	31	32	33	34	35	36	37	38	39
1	0.21419E-06	0.70734E-10	0.14390E-02	0.43658E-07	0.22766E-11	0.25782E-07	0.71182E-07	0.17299E-06	0.95030E-02	
2	0.94837E-02	0.32456E-06	0.67532E-07	0.16923E-02	0.64767E-06	0.18477E-05	0.23738E-04	0.95030E-02	0.10060E-06	
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	-0.25962	-0.55009E-05	0.59207E-05	0.59922E-02	0.95460E-05	0.47687E-04	0.60010E-03	0.23919	0.25090E-05	
5	0.63399E-05	0.13451E-08	0.39224E-01	-0.15584E-05	0.89647E-09	0.10873E-05	0.21410E-05	0.33499E-05	0.23626	
6	0.62814E-07	0.16387E-05	0.36914E-05	0.10877E-04	0.28578E-03	-1.0000	-0.96482	0.61941E-05	0.10022E-07	
7	-0.41806	-0.18409E-04	0.18680E-04	1.0000	-0.44709E-04	0.12276E-03	0.13433E-02	0.54460	0.57681E-05	
8	0.84408E-05	0.41826E-07	-1.0000	-0.20101E-04	0.10645E-07	0.38420E-05	0.74727E-05	0.10163E-04	0.54457	
9	-0.22077E-07	0.18196E-05	0.70509E-06	0.23154E-04	0.23749E-03	0.93614	1.0000	-0.47688E-05	0.65863E-08	
10	-1.0000	-0.35499E-04	0.62997E-05	0.24452	0.74495E-04	0.19388E-03	0.24994E-02	1.0000	0.10586E-04	
11	0.22675E-04	0.10304E-07	0.21885	-0.38643E-05	0.99016E-09	0.26520E-05	0.76087E-05	0.18148E-04	-1.0000	
12	0.20115E-06	0.40212E-05	0.89238E-07	0.11831E-05	0.10200E-04	0.56036E-01	0.29726E-01	0.12100E-06	0.13114E-09	
13	-0.85428E-06	0.23063E-09	0.64652E-02	0.19949E-06	0.13262E-09	0.13106E-06	0.16422E-06	0.16042E-06	0.84100E-02	
14	-0.37911E-01	0.11561E-05	0.34132E-06	0.16508E-02	0.17672E-05	0.16563E-05	0.19816E-04	0.81765E-02	0.86624E-07	
15	0.19933E-06	-1.0000	0.67518E-08	0.95973E-05	0.99351	0.30125E-04	0.32411E-05	0.14268E-07	0.22357E-12	
16	-0.40967E-06	0.54969E-08	0.13083	0.19747E-05	0.13928E-08	0.42633E-06	0.51975E-06	0.10362E-06	0.95088E-02	
17	-0.23135E-01	0.94020E-06	0.10285E-05	0.11679	-0.73190E-05	0.36253E-05	0.21917E-04	0.95110E-02	0.99479E-07	
18	-0.64389E-07	0.99287	0.89966E-08	0.98817E-05	-1.0000	-0.28376E-04	0.43803E-05	0.41320E-08	0.22901E-13	
19	-0.20574E-06	0.13690E-09	0.34768E-02	0.12159E-06	0.46949E-10	0.40679E-07	0.95850E-07	0.17229E-06	0.95030E-02	
20	-0.87105E-02	0.28594E-06	0.33462E-06	0.18837E-01	0.10397E-05	0.21738E-05	0.23425E-04	0.95033E-02	0.10058E-06	
21	-0.10780E-10	0.12623E-02	0.10662E-10	0.29464E-08	0.63554E-03	0.11491E-07	0.76956E-07	0.90330E-10	0.23525E-15	
22	0.25962	0.55009E-05	0.59207E-05	0.59922E-02	0.95460E-05	0.47687E-04	0.60010E-03	0.23919	-0.25090E-05	
23	0.63399E-05	0.13451E-08	0.39224E-01	-0.15584E-05	0.89647E-09	0.10873E-05	0.21410E-05	0.33499E-05	0.23626	
24	-0.62814E-07	0.16387E-05	0.36914E-05	0.10877E-04	-0.28578E-03	1.0000	0.96482	-0.61941E-05	0.10022E-07	
25	0.41806	0.18409E-04	0.18680E-04	-1.0000	0.44709E-04	0.12276E-03	0.13433E-02	0.54460	0.57681E-05	
26	0.84408E-05	0.41826E-07	-1.0000	-0.20101E-04	0.10645E-07	0.38420E-05	0.74727E-05	0.10163E-04	0.54457	
27	0.22077E-07	0.18196E-05	0.70509E-06	0.23154E-04	0.23749E-03	0.93614	-1.0000	0.47688E-05	0.65863E-08	
28	1.0000	0.35499E-04	0.62997E-05	0.24452	-0.74495E-04	0.19388E-03	0.24994E-02	-1.0000	-0.10586E-04	
29	0.22675E-04	0.10304E-07	0.21885	-0.38643E-05	0.99016E-09	0.26520E-05	0.76087E-05	0.18148E-04	-1.0000	
30	-0.20115E-06	0.40212E-05	0.89238E-07	0.11831E-05	0.10200E-04	0.56036E-01	0.29726E-01	0.12100E-06	0.13114E-09	
31	-0.85428E-06	0.23063E-09	0.64652E-02	0.19949E-06	0.13262E-09	0.13106E-06	0.16422E-06	0.16042E-06	0.84100E-02	
32	0.37911E-01	0.11561E-05	0.34132E-06	0.16508E-02	0.17672E-05	0.16563E-05	0.19816E-04	0.81765E-02	0.86624E-07	
33	0.19933E-06	-1.0000	0.67518E-08	0.95973E-05	0.99351	0.30125E-04	0.32411E-05	0.14268E-07	0.22357E-12	
34	-0.40967E-06	0.54969E-08	0.13083	0.19747E-05	0.13928E-08	0.42633E-06	0.51975E-06	0.10362E-06	0.95088E-02	
35	0.23135E-01	0.94020E-06	0.10285E-05	0.11679	-0.73190E-05	0.36253E-05	0.21917E-04	0.95110E-02	0.99479E-07	
36	-0.64389E-07	0.99287	0.89966E-08	0.98817E-05	-1.0000	-0.28376E-04	0.43803E-05	0.41320E-08	0.22901E-13	
37	-0.20574E-06	0.13690E-09	0.34768E-02	0.12159E-06	0.46949E-10	0.40679E-07	0.95850E-07	0.17229E-06	0.95030E-02	
38	0.87105E-02	0.28594E-06	0.33462E-06	0.18837E-01	0.10397E-05	0.21738E-05	0.23425E-04	0.95033E-02	0.10058E-06	
39	-0.10780E-10	0.12623E-02	0.10662E-10	0.29464E-08	0.63554E-03	0.11491E-07	0.76956E-07	0.90330E-10	0.23525E-15	

ROTOR DAMPING MATRIX

TABLE 10.10 (Cont'd)

APY-CRYA

ROW	COLUMN	1	2	3	4	5	6	7	8	9	10
1	C.C		-0.34884E-01	-0.11478E-04	0.62342E-C2	1839.0	-0.95430E-04	10779.	-0.47031E-01	-0.17644E-05	0.12575E-04
2	C.C		20431.	-0.15104E-04	-1878.2	0.63323E-02	-0.60481E-01	0.32693E-02	11118.	-0.10174E-01	-29.2294
3	C.C		0.21809E-02	373739.	-0.59604E-01	0.10396	0.10250E-01	0.82285	0.47153	0.33562E-02	0.33562E-02
4	C.C		1789.1	-0.73161E-01	406.18	-1.2345	4.8270	-8.2540	-1248.5	-0.14603	-58.627
5	C.C		0.11155E-01	0.97475E-01	-1.5162	381.52	-3.8700	1149.7	16.050	0.38186	-0.41046E-01
6	C.C		-0.41236E-01	-0.76514	1.1200	-0.74337	1133.8	-5.1696	-11.783	1.5705	0.16105E-01
7	C.C		0.18931E-01	0.27054	-2.2961	1149.0	-7.6248	51891.	24.229	2.0806	0.23520
8	C.C		-1310.4	0.27229	-1239.9	8.1252	-32.447	54.367	52181.	0.76330	-691.36
9	C.C		0.63254E-01	1959.8	-0.11084	0.19710	0.64806E-02	1.6706	0.92324	7252.5	0.20296E-02
10	C.C		905.52	-0.77035E-01	-58.643	-0.48290E-01	0.10425	-0.34618	-691.78	0.18863	395.27
11	C.C		-0.78728E-02	-0.90358E-02	0.86904E-01	-0.17188	0.28848	656.15	-0.80445	-0.56133E-01	0.57467E-02
12	C.C		-0.10412E-02	-0.43386	0.26130	0.17188	-198.47	1.2482	2.7982	0.87692	-0.58893E-02
13	C.C		-0.11108	-0.28442	0.87276	-686.39	-2.6102	6483.0	-8.8242	-0.45638	0.12677E-01
14	C.C		-1920.0	-1.0499	654.99	0.82588	-2.1250	6.1320	6383.1	1.5965	1190.6
15	C.C		-0.14457E-01	594.17	-0.40532E-01	0.32840E-02	-0.71969E-01	0.65657E-01	0.38907	-1275.3	0.26486E-02
16	C.C		6621.7	0.91070E-01	-8.5245	0.64313E-02	-0.77915E-01	-0.83970	-114.61	0.16586	-304.60
17	C.C		-0.46193E-01	-0.80404E-02	0.14797	-18.027	-0.38097	174.47	1.4090	0.15100E-01	-0.64018E-03
18	C.C		0.20908E-01	0.52781	-0.81903E-01	0.55429E-01	-111.16	0.35062	0.81077	-1.0666	0.83528E-03
19	C.C		-0.27620	-0.26546	-2.3217	-949.32	-5.2718	8232.2	7.3837	0.32283	0.75428
20	C.C		-19422.	-3.4674	471.98	-0.72199	2.4643	8.4766	5313.5	6.6247	11467.
21	C.C		0.32561E-01	78659.	0.44608	-0.66188	-0.12022	-5.3749	-2.2263	-938.97	-0.11063
22	C.C		-1789.1	-0.73164E-01	59.292	-1.2345	4.8204	-8.2562	-251.06	-0.14463	4.5261
23	C.C		0.13445E-01	-0.97478E-01	1.5112	-82.624	3.8698	-315.40	-16.014	-0.38186	0.41056E-01
24	C.C		0.40497E-01	0.76498	1.1157	-0.74327	3.3083	-5.1722	-11.751	1.5764	0.15935E-01
25	C.C		0.57943E-01	-0.27053	2.2944	-314.71	7.6311	-1682.3	-24.187	-2.0806	-0.23567
26	C.C		1311.1	0.27230	-242.46	8.1231	-32.446	54.356	1268.5	0.73117	-3.7301
27	C.C		0.62464E-01	-1959.8	0.11089	-0.19710	-0.16263E-01	-1.6706	-0.91395	78.933	-0.73961E-02
28	C.C		-905.50	-0.77034E-01	4.5144	-0.48267E-01	0.10106	-0.34536	-4.1583	0.19464	1.4408
29	C.C		-0.75134E-02	0.90363E-02	-0.85871E-01	-6.7947	-0.28986	-9.3448	0.79341	0.56131E-01	-0.60704E-02
30	C.C		0.10643E-02	-0.43389	-0.26059	0.17186	-0.83974	1.2486	2.7989	0.87671	-0.33433E-02
31	C.C		-0.95954E-01	0.28446	-0.84466	-62.785	-2.5894	-156.35	8.6315	0.45642	-0.93754E-02
32	C.C		1920.3	-1.0499	-39.021	0.82554	-2.1598	6.1130	84.239	1.6489	-6.4913
33	C.C		-0.14780E-01	-594.16	0.40869E-01	0.32844E-02	0.82343E-01	0.65659E-01	-0.39894	15.516	0.25975E-02
34	C.C		-6621.7	0.91070E-01	1.3946	0.64428E-02	-0.65868E-01	-0.84013	3.0026	-0.19202	1.2054
35	C.C		-0.45754E-01	0.80410E-02	0.13721	-6.7106	0.37781	8.4221	-1.3470	-0.15097E-01	0.93215E-03
36	C.C		-0.20720E-01	0.52780	-0.81162E-01	0.55421E-01	-0.20273	0.35089	0.83659	-1.0812	-0.22294E-02
37	C.C		-0.25404	0.26548	2.3534	-324.20	5.3958	197.92	-7.5548	-0.32274	-0.73763
38	C.C		19422.	-3.4674	-78.893	-0.72144	2.3582	8.4813	-61.397	7.0792	-42.741
39	C.C		0.33260E-01	-78659.	-0.45071	0.66188	-0.21683	5.3749	2.2615	72.911	0.10240

TABLE 10.10 (Cont'd)

ROW	COLUMN	11	12	13	14	15	16	17	18	19	20
1	31.766	0.32617E-04	387.57	-0.14905E-02	-0.50569E-06	0.68685E-04	19.443	-0.54759E-06	1085.8	-0.24036E-02	
2	-0.66490E-03	0.30445E-01	-0.10452	353.36	0.38629E-02	-4.2665	-0.25290E-02	-0.78564E-02	-0.95029E-01	353.50	
3	-0.59087E-02	0.78737E-02	0.20284	-0.11435	-0.299.40	0.62656E-03	-0.10731E-02	-0.86078E-02	-0.14068E-01	-0.96216E-02	
4	0.13864	-2.5108	2.1081	655.68	-0.41374E-01	-8.5163	-0.25470E-01	0.67265	-1.1577	471.14	
5	-58.152	2.0034	-689.33	-0.47255E-01	-0.13184	0.81746E-02	-18.140	-0.53095	-949.93	-0.49277	
6	0.91218E-01	-199.40	1.2077	0.75866E-01	-1.2921	0.41880E-02	-0.17631E-01	-110.46	-0.79090	-0.87521E-01	
7	655.70	4.2881	6490.7	2.4692	-0.94649	-0.28579	173.68	-1.3045	8215.6	11.076	
8	-0.77597	17.304	-14.482	6370.0	0.21161	-114.68	0.40710E-01	-4.8349	2.4004	5323.9	
9	-0.67119E-02	0.17044E-02	-0.37484	-0.24303	-1276.0	0.48476E-02	-0.38146E-02	-0.23498E-03	-0.92165E-01	-0.14522	
10	0.19308E-02	0.50374E-01	0.79488E-01	1190.9	-0.13760	-304.60	0.14795E-02	0.11487E-01	0.56843E-01	11468.	
11	435.85	-0.15798	-1365.4	0.17266	0.24926E-01	-0.81549E-02	-357.26	0.46115E-01	-13902.	0.30679	
12	-0.17209E-01	1120.4	-0.27069	-0.19852E-01	-0.70657	-0.35363E-03	0.37594E-02	-715.91	0.17266	-0.10779E-02	
13	-1365.6	-1.4419	52817.	0.12932	0.85229E-01	-0.41063E-02	2181.7	0.42690	73304.	0.28680	
14	0.78066E-01	0.99744	-2.3959	51964.	-1.4819	-2006.5	-0.13416	-0.21366	-5.2031	65778.	
15	-0.13267E-02	0.25038E-01	0.69407E-02	-0.80135E-02	7340.7	-0.22830E-02	0.66353E-03	-0.23657E-03	0.21999E-01	0.88057E-01	
16	-0.19288E-01	0.35348E-01	0.40415	-2006.7	0.14111	378.04	-0.54599E-02	-0.71251E-02	-0.19771	-12858.	
17	-357.27	0.20705	2181.7	-0.49418E-01	-0.10612E-01	0.25593E-03	421.28	-0.59821E-01	16976.	-0.19375E-02	
18	-0.71839E-02	-715.67	-0.96630E-01	-0.11362E-01	0.84626	-0.10106E-02	0.15853E-02	2485.0	0.68660E-01	0.35331E-01	
19	-13902.	2.4511	73316.	-7.6958	-0.25592	-0.29180	16977.	-0.51277	0.84635E 06	13.504	
20	0.28129	-1.2083	7.9950	65786.	-5.4069	-12858.	0.15577	0.28360	5.8978	0.58711E 06	
21	0.14943	0.16534	0.18196	1.3822	-5137.6	0.37978E-01	-0.10861	-0.95082E-01	-4.4011	-1.7790	
22	0.13829	-2.5105	2.1121	-38.362	-0.41956E-01	1.4064	-0.25049E-01	0.67275	-1.1357	-79.891	
23	-6.6098	-2.0034	-59.854	0.47510E-01	0.13184	-0.82150E-02	-6.5954	0.53097	-323.58	0.49424	
24	0.81531E-01	-1.7813	1.1998	0.75257E-01	-1.2973	0.45911E-02	-0.18106E-01	0.50593	-0.81023	-0.92186E-01	
25	-8.9047	-4.2902	-164.17	-2.4412	0.94649	0.28529	9.1868	1.3045	215.39	-11.060	
26	-0.77495	17.319	-14.468	70.968	0.23313	2.9218	0.40167E-01	-4.8483	2.3735	-51.039	
27	0.67111E-02	0.30394E-02	0.37485	0.19769	16.216	0.11290E-02	0.38156E-02	0.19905E-02	0.92211E-01	-0.67770E-01	
28	0.20474E-02	0.43504E-01	0.78644E-01	-6.2572	-0.14415	1.1980	0.13843E-02	0.72828E-02	0.53667E-01	-42.469	
29	-2.6852	0.15886	-11.934	-0.18086	-0.24925E-01	0.86071E-02	-4.9049	-0.46427E-01	-164.41	-0.32290	
30	-0.17294E-01	0.46622	-0.26897	0.42442E-02	-0.70293	-0.26209E-02	0.38743E-02	-0.13907	0.17724	0.88414E-01	
31	-11.670	1.4324	-101.56	-0.71376E-01	-0.85242E-01	0.52234E-03	-10.503	-0.42503	-638.69	-0.15631	
32	0.85622E-01	1.0686	-2.4784	41.146	-1.5453	-2.4127	-0.14203	-0.25892	-5.4989	148.97	
33	0.13270E-02	0.32248E-01	-0.69445E-02	0.55802E-01	3.3736	-0.30488E-02	-0.66364E-03	0.37626E-02	-0.21989E-01	0.11430	
34	-0.19570E-01	0.27094E-01	0.40158	-2.5954	0.15183	4.3957	-0.51685E-02	-0.34873E-02	-0.18501	-40.918	
35	-4.8980	-0.20482	-10.517	0.25195E-01	0.10611E-01	0.31735E-04	-38.449	0.58884E-01	-392.93	-0.15150E-02	
36	-0.71926E-02	0.10381	-0.96084E-01	-0.13065E-01	0.89528	0.96660E-03	0.16087E-02	-0.29470E-01	0.69593E-01	-0.51710E-01	
37	-164.44	-2.5276	7.9476	-650.95	0.25589	0.27169	-393.16	0.53950	-19147.	-12.813	
38	0.28371	-0.93490	7.9550	155.99	-5.8076	-40.924	0.15247	0.10900	5.7338	1521.2	
39	-0.14943	0.22511	-0.18197	-1.4674	20.563	-0.29351E-01	0.10861	-0.11574	4.4009	1.4529	

TABLE 10.10 (Cont'd)

ROW	COLUMN	21	22	23	24	25	26	27	28	29	30
1	0.16114E-05	-0.61869E-02	1639.0	0.97881E-04	10779.	0.46455E-01	0.22713E-05	-0.11190E-04	31.767	-0.36644E-04	
2	0.29573E-03	1878.2	0.61753E-02	0.60473E-01	0.34127E-02	-1.1118.	-0.10165E-01	29.294	-0.64753E-03	-0.30431E-01	
3	-774.72	-0.59590E-01	0.10396	0.10261E-01	-0.82320	0.47169	2912.2	0.33478E-02	0.59027E-02	0.78644E-02	
4	-0.71306E-01	59.292	1.2344	4.8204	8.2574	-251.07	0.14463	4.5259	-0.13826	-2.5105	
5	-0.23373E-02	-1.5112	-82.624	-3.8698	-315.41	16.011	-0.38186	-0.40975E-01	-6.6094	2.0034	
6	0.61581	1.1157	0.74327	3.3091	5.1721	-11.751	-1.5768	0.15935E-01	-0.81533E-01	-1.7814	
7	0.17322	-2.2939	-314.71	-7.6312	-1682.4	24.150	-2.0805	0.23658	-8.9002	4.2902	
8	-0.36425	-242.47	-8.1212	-32.447	-54.349	1268.5	-0.73111	-3.7261	0.77436	17.319	
9	-932.17	-0.11089	-0.19710	0.16255E-01	-1.6706	0.91397	78.935	0.73936E-02	0.67087E-02	-0.30350E-02	
10	0.59689E-01	4.5143	0.48252E-01	0.10107	0.34513	-4.1576	-0.19464	1.4410	-0.20355E-02	0.43504E-01	
11	-0.41111E-02	0.85873E-01	-6.7945	0.28986	-9.3489	-0.79376	0.56136E-01	0.60675E-02	-2.0856	-0.15885	
12	0.33541	-0.26059	-0.17186	-0.84043	-1.2486	2.7989	-0.87673	-0.33430E-02	0.17293E-01	0.46649	
13	0.66022E-01	0.84462	-62.787	2.5895	-156.36	-8.6333	0.45646	0.95034E-02	-11.666	-1.4324	
14	0.76271	-39.021	-0.82493	-2.1598	-6.1106	84.250	-1.6489	-6.4922	-0.85767E-01	1.0686	
15	-5139.0	-0.40867E-01	0.32851E-02	0.82335E-01	0.65631E-01	0.39895	15.506	-0.25981E-02	0.13264E-02	0.32247E-01	
16	0.68581E-01	1.3948	-0.64445E-02	0.65868E-01	0.84001	3.0026	0.19202	1.2049	0.19567E-01	0.27093E-01	
17	0.46150E-02	-0.13721	-6.7106	-0.37781	8.4252	-1.3471	-0.15099E-01	-0.92498E-03	-4.8976	0.20482	
18	-0.39692	-0.81163E-01	-0.55421E-01	-0.20251	-0.35087	0.83656	-0.32281	-0.22296E-02	0.71937E-02	0.10366	
19	0.12570	-2.3534	-324.19	-5.3958	198.05	7.5615	-0.70793	0.73790	-164.42	2.5275	
20	2.5639	-78.900	0.72117	2.3582	-8.4745	-61.409	-7.0793	-42.718	-0.28360	-0.93489	
21	87528.	0.45071	0.66188	0.21683	5.3750	-2.2615	72.919	-0.10240	-0.14943	-0.22511	
22	0.71469E-01	406.18	1.2343	4.8220	8.2548	-1248.5	0.14603	-58.627	-0.13861	-2.5108	
23	0.23377E-02	1.5162	381.53	3.8700	1149.7	-16.047	0.38186	0.40964E-01	-58.152	-2.0034	
24	0.61957	1.1200	0.74337	1133.8	5.1695	-11.782	-1.5702	0.16105E-01	-0.81220E-01	-199.40	
25	-0.17322	2.2949	1149.0	7.6249	51891.	-24.183	2.0805	-0.23614	655.70	-4.2881	
26	-0.37193	-1239.9	-8.1236	-32.447	-54.374	52181.	-0.76526	-691.35	0.77528	17.304	
27	66.197	0.11084	0.19710	-0.64739E-02	1.6706	-0.92326	7252.5	-0.20278E-02	-0.67102E-02	0.16988E-02	
28	0.63861E-01	-58.643	0.48283E-01	0.10425	0.34626	-691.78	-0.18863	395.27	-0.19153E-02	-0.50373E-01	
29	0.41110E-02	0.86924E-01	-57.967	-0.28849	656.15	0.80498	-0.56137E-01	-0.57410E-02	435.85	0.15798	
30	0.32862	-0.26131	-0.17188	-198.47	-1.2482	2.7981	-0.87705	-0.54890E-02	0.17208E-01	1120.4	
31	-0.66025E-01	-0.87322	-686.39	-2.6103	6482.9	8.8307	-0.45645	-0.12749E-01	-1365.6	1.4418	
32	0.80602	654.99	-0.82575	-2.1250	-6.1351	6383.1	-1.5965	1190.6	-0.78160E-01	0.99743	
33	21.404	0.40529E-01	-0.32852E-02	0.71964E-01	-0.65633E-01	-0.38907	-1275.3	-0.26478E-02	-0.13260E-02	-0.25033E-01	
34	-0.71168E-01	-8.5244	-0.64198E-02	0.77915E-01	0.83970	-114.61	0.16586	-304.60	0.19283E-01	0.35348E-01	
35	-0.46109E-02	0.14796	-18.027	0.38098	174.47	-1.4090	0.15101E-01	0.63107E-03	-357.27	-0.20705	
36	-0.42754	-0.81910E-01	-0.55430E-01	-111.16	-0.35060	0.81074	1.0666	0.83505E-03	0.71850E-02	-715.67	
37	-0.12571	2.3214	-949.32	5.2718	8232.1	-7.3858	0.32286	-0.75465	-1390.2	-2.4511	
38	2.7831	471.98	0.72125	2.4643	-8.4741	5313.4	-6.6248	11467.	-0.28110	-1.2083	
39	1919.6	-0.44608	-0.66188	0.12022	-5.3750	2.2263	-938.98	0.11063	0.14943	-0.16534	

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TABLE 10.10 (Cont'd)

ROW	COLUMN	31	32	33	34	35	36	37	38	39
1		387.57	C.14663E-02	C.59205E-02	-C.67252E-04	19.443	0.29808E-05	1085.8	0.23437E-02	-0.15406E-05
2	-C.13455		0.38641E-02	0.38641E-02	4.2664	-0.25272E-02	0.78473E-02	-0.94953E-01	-353.48	0.28108E-03
3	C.20299		-0.11426	299.37	-0.66099E-03	0.10552E-02	-0.86020E-02	0.14408E-01	-0.97698E-02	774.73
4	-2.1126		-38.358	0.49519E-01	1.4063	0.25035E-01	0.67275	1.1348	-79.899	-0.71471E-01
5	-59.854		-C.46910E-01	0.13186	0.81894E-02	-6.5950	-0.53097	-323.58	-0.49389	0.23291E-02
6	-1.1992		0.75254E-01	1.2980	0.45899E-02	0.18106E-01	0.50607	0.81021	-0.92086E-01	-0.62004
7	-164.14		2.4523	0.94644	-0.28552	9.1788	-1.3045	215.04	11.062	-0.17318
8	14.470		70.934	-0.23320	2.9175	-0.39381E-01	-4.8484	-2.3326	-50.891	0.37197
9	C.37485		-C.19765	16.229	-0.11320E-02	0.38127E-02	0.19223E-02	0.92262E-01	0.67706E-01	66.187
10	-C.78598E-01		0.14416	0.14416	1.1979	-0.14049E-02	0.72817E-02	0.54603E-01	-42.466	-0.63866E-01
11	-11.930		0.18101	-0.24935E-01	-0.86020E-02	-4.9047	0.46423E-01	-164.39	0.32277	0.41196E-02
12	C.26897		0.42442E-02	0.79329	-0.26206E-02	-0.38739E-02	-0.13983	-0.17722	0.88389E-01	-0.32894
13	-101.59		C.72593E-01	-0.85324E-01	-0.55567E-03	-10.507	0.42501	-638.84	0.15802	-0.65967E-01
14	2.4747		41.181	1.5453	-2.4157	0.14221	-0.25892	5.5090	149.08	-0.80602
15	-0.69252E-02		-C.55806E-01	3.3768	0.30472E-02	0.66634E-03	-0.37631E-02	-0.22053E-01	-0.11432	21.420
16	0.40163		-2.5965	-0.15184	4.3957	0.51702E-02	-0.34865E-02	0.18494	-40.923	0.71171E-01
17	-10.521		-0.25219E-01	0.10615E-01	-0.37011E-04	-38.449	-0.58882E-01	-392.94	0.16597E-02	-0.46150E-02
18	0.96077E-01		-C.13065E-01	-0.89567	0.96668E-03	-0.16089E-02	-0.28285E-01	-0.69609E-01	-0.51716E-01	0.42785
19	-651.07		-7.9486	C.25607	-0.27188	-393.17	-0.53941	-19147.	12.818	-0.12587
20	-7.9569		156.01	5.8078	-40.934	-0.15248	0.10897	-5.7268	1521.3	-2.7832
21	-0.18199		1.4674	20.559	0.29354E-01	0.10862	0.11574	4.4009	-1.4529	1919.5
22	-2.1087		555.68	0.41369E-01	-8.5163	0.25453E-01	0.67266	1.1567	471.14	-0.71309E-01
23	-689.33		0.46602E-01	-0.13186	-0.81474E-02	-18.141	0.53095	-949.94	0.49235	-0.23294E-02
24	-1.2077		0.75852E-01	1.2923	0.41867E-02	0.17630E-01	-110.46	0.79087	-0.87421E-01	-0.61603
25	6490.7		-2.4797	-0.94644	0.28600	173.69	1.3046	8215.9	-11.076	0.17318
26	14.483		6370.0	-0.21167	-114.69	-0.39999E-01	-4.8350	-2.3623	5324.0	0.36429
27	-0.37486		C.24302	-1276.0	-0.48442E-02	-0.38112E-02	0.23800E-03	-0.92191E-01	0.14526	-932.16
28	-0.79349E-01		1190.9	0.13760	-304.61	-0.14993E-02	0.11485E-01	-0.57767E-01	11468.	-0.59693E-01
29	-1365.4		-0.17284	0.29935E-01	0.81481E-02	-357.26	-0.46112E-01	-13902.	-0.30659	-0.41194E-02
30	C.27069		-C.19862E-01	0.70702	-0.35328E-03	-0.37591E-02	-715.92	-0.17264	-0.11024E-02	-0.33577
31	52817.		-C.12264	0.85317E-01	0.41013E-02	2181.7	-0.42686	73304.	-0.28682	-0.65972E-01
32	2.3940		51964.	1.4818	-2006.5	0.13428	-0.21366	5.2104	65778.	-0.76272
33	0.69255E-02		0.80134E-02	7340.7	0.22845E-02	0.66618E-03	0.23331E-03	0.22060E-01	-0.88037E-01	-5139.0
34	0.40413		-2006.7	-0.14111	378.04	0.54636E-02	-0.71245E-02	0.19774.	-12858.	0.68584E-01
35	2181.7		0.49471E-01	-0.10616E-01	-0.25043E-03	421.28	0.59819E-01	16976.	0.17768E-02	0.46192E-02
36	0.96623E-01		-0.11362E-01	-0.84661	-0.10105E-02	-0.15856E-02	2485.0	-0.68676E-01	0.35324E-01	0.39721
37	73316.		7.6982	-0.25610	0.29201	16977.	0.51269	0.84635E 06	-13.510	0.12587
38	-7.9944		65786.	5.4072	-12858.	-0.15585	0.28358	-5.8939	0.58711E 06	-2.5640
39	0.18198		-1.3822	-5137.6	-0.37981E-01	-0.10862	0.95084E-01	-4.4011	1.7790	87528.

TABLE 10.11 SAMPLE OUTPUT; "T," BASELINE HUB

RUN NUMBER=	I	HUB T CONF.	FUN NO.	1.
DD(100)	=	1.000000		HUB CALCULATIONS
DD(104)	=	-27.79555		YHG(1)
DD(107)	=	-63.00000		YHC1(1)
DD(110)	=	-122.0000		YHC2(1)
DD(113)	=	-175.1500		YHC3(1)
DD(116)	=	-156.5000		YHE(1)
DD(119)	=	1735325.		CI1(1,1)
DD(123)	=	2312554.		CI1(2,2)
DD(127)	=	252228.8		CI1(3,3)
DD(128)	=	1733430.		CI2(1,1)
DD(132)	=	1829283.		CI2(2,2)
DD(136)	=	98517.31		CI2(3,3)
DD(137)	=	1644065.		CI3(1,1)
DD(141)	=	2934929.		CI3(2,2)
DD(145)	=	485755.5		CI3(3,3)
DD(146)	=	741.8020		CM1
DD(147)	=	536.5137		CM2
DD(148)	=	3881.588		CM3
DD(149)	=	0.5599558E-02		DAMPH
DD(51)	=	15.82300		AREA
DD(52)	=	13446.00		AI
DD(53)	=	4038500.		GBE
DD(54)	=	2.1105000E 08		BE
DD(55)	=	26892.00		PMJ
DD(90)	=	7.000000		IRIT
DD(92)	=	0.1000000E-04		TPHMF
DD(95)	=	10.00000		TK7H
DD(96)	=	10.00000		TCH
DD(97)	=	10.00000		TK6HG
DD(98)	=	10.00000		TCHG

HUB STIFFNESS MATRIX

TABLE 10.11 (Cont'd)

APRY-K7H

ROW	COLUMN	1	2	3	4	5	6	7	9	10
1	C.75359E 07	C.0	C.0	C.0	C.0	0.0	0.0	-0.28160E 07	C.0	0.0
2	C.0	0.32704E 06	C.0	0.0	0.0	0.0	0.30578E 07	0.0	-57286.	0.0
3	C.0	C.0	0.32704E 06	C.0	0.0	-0.30578E 07	0.0	0.0	0.0	0.0
4	C.0	C.0	C.0	0.0	0.34209E 08	C.0	0.0	0.0	0.0	-0.12783E 08
5	C.0	C.0	-0.30578E 07	C.0	-0.30578E 07	C.17788E 09	C.0	0.0	-0.16899E 07	0.0
6	C.0	0.30578E 07	C.0	0.0	0.0	0.0	0.17788E 09	0.0	0.16899E 07	0.0
7	-0.28160E 07	C.0	C.0	C.0	0.0	0.0	0.0	0.76317E 07	C.0	0.0
8	C.0	-57286.	C.0	0.0	0.0	0.0	0.16899E 07	0.0	0.34380E 06	0.0
9	C.0	C.0	-57286.	C.0	0.0	-0.16899E 07	0.0	0.0	0.34380E 06	0.0
10	C.0	C.0	C.0	0.0	-0.12783E 08	C.0	0.0	0.0	0.0	0.34643E 08
11	C.0	C.0	0.0	0.0	0.0	0.33235E 08	0.0	0.0	0.0	0.0
12	0.0	-0.16899E 07	C.0	C.0	C.0	0.0	0.33235E 08	0.0	-0.32524E 07	C.0
13	C.0	C.0	C.0	C.0	C.0	0.0	0.0	-0.48157E 07	0.0	0.0
14	C.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.28651E 06	0.0
15	C.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
16	C.0	0.0	0.0	0.0	C.0	C.0	0.0	0.0	-0.28651E 06	0.0
17	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.21861E 08
18	C.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	-0.10271E 08	0.0

APRY-K7H

ROW	COLUMN	11	12	13	14	15	16	17	18
1	C.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	C.0	-0.16899E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.16899E 07	C.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.33235E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	C.0	0.33235E 08	0.0	0.0	0.0	0.0	0.0	0.0	0.0
7	C.0	0.0	-0.48157E 07	0.0	0.0	0.0	0.0	0.0	0.0
8	0.0	-0.32524E 07	0.0	C.0	-0.28651E 06	0.0	0.0	0.0	-0.10271E 08
9	0.32524E 07	0.0	0.0	0.0	0.0	-0.28651E 06	0.0	0.10271E 08	0.0
10	0.0	C.0	0.0	0.0	0.0	0.0	-0.21861E 08	0.0	0.0
11	0.18014E 09	0.0	0.0	0.0	0.0	-0.49423E 07	0.0	0.14876E 09	0.0
12	0.0	0.18014E 09	0.0	0.0	0.49423E 07	0.0	0.0	0.0	0.14876E 09
13	C.0	0.0	0.0	0.48157E 07	0.0	0.0	0.0	0.0	0.0
14	0.0	0.49423E 07	0.0	0.0	0.28651E 06	0.0	0.0	0.0	0.10271E 08
15	-0.49423E 07	0.0	0.0	0.0	0.0	0.28651E 06	0.0	-0.10271E 08	0.0
16	0.0	0.0	0.0	0.0	0.0	0.0	0.21861E 08	0.0	0.0
17	0.14876E 09	0.0	0.0	0.0	0.0	-0.10271E 08	0.0	0.39665E 09	0.0
18	C.0	0.14876E 09	0.0	0.0	0.0	0.0	0.0	0.0	0.39665E 09

HUE DAMPING MATRIX

TABLE 10.11 (cont'd)

ARRY-CH

ROW COLUMN	1	2	3	4	5	6	7	8	9	10
1	1459.8	-0.14182E-04	0.48545E-04	0.25974E-07	0.46197E-03	0.19926E-03	-270.47	0.11271E-04	C.57674E-05	0.41705E-08
2	-0.92206E-03	289.98	-0.33141E-01	0.44794E-02	0.22745	1631.3	0.20631E-02	-45.658	C.15926E-01	0.71949E-02
3	-0.82522E-04	-0.37401E-02	304.21	C.86874E-03	-2399.9	1.3285	-0.31671E-02	C.20960E-01	-32.652	-0.87556E-02
4	-C.25224E-04	-0.14846	0.62892E-01	C.15036E-06	0.36099	-4.1539	0.12809E-03	-0.29020	-0.11532	-32.676.
5	-C.74243E-02	-0.10580	-2390.9	C.47509E-01	0.33469E-06	-16.973	0.26368E-01	C.11601E-01	-1619.6	0.15517
6	-C.63217E-03	1630.4	1.1202	-0.24887	-12.060	0.13653E-06	0.63033E-02	1370.5	-0.43606E-01	0.31421
7	-270.47	-0.12834E-04	0.24715E-03	0.13556E-06	0.23367E-02	0.45626E-03	1231.0	0.51901E-04	0.32607E-04	0.35748E-07
8	-0.14455E-03	-45.665	0.23367E-01	0.12877E-01	0.19940	1070.3	0.55958E-03	298.84	-C.11151E-01	0.81955E-02
9	-C.12842E-03	0.13684E-01	-32.666	-0.14664E-02	-1619.5	0.11062	0.45781E-03	0.10149E-01	241.6C	0.34825E-02
10	C.86474E-05	0.17981	-0.64737E-01	-32677.	1.3424	3.5823	-0.42914E-04	C.12207	-0.11806	0.13711E-06
11	C.15676E-01	2.23941	1365.6	-0.59032E-01	56283.	16.376	-0.72159E-01	0.31996E-01	1517.7	0.26595
12	0.41795E-02	-664.75	0.43641E-01	0.40269E-01	0.40856	12465.	-0.17692E-01	-811.48	-0.51193E-01	0.48342E-01
13	-144.92	-0.45250E-03	0.21293E-03	0.43113E-06	0.17851E-02	0.57963E-02	-708.81	0.18818E-03	-C.38291E-04	0.30343E-06
14	C.21904E-04	0.60090	0.19552E-02	-C.14664E-01	0.43794E-01	79.488	0.16464E-05	-140.61	0.51857E-02	0.12832E-01
15	-0.81968E-04	-0.88455E-02	-14.663	0.75719E-03	-1281.0	-0.37416	0.36159E-03	C.10249E-02	-176.94	-0.29545E-02
16	C.93636E-06	0.17243	0.11658	-7932.3	-0.92208	-3.7557	-0.39672E-05	0.54956E-01	-C.24586	-62785.
17	-C.35282E-02	-0.36183	510.73	0.13519	39146.	-8.2102	0.19769E-01	0.52379	5828.5	-0.15850
18	-C.49533E-02	-138.51	0.54177	0.75766	-6.2347	4099.9	0.27637E-01	-4281.6	0.16431	-0.333602

ARRY-CH

ROW COLUMN	11	12	13	14	15	16	17	18
1	-0.34960E-03	-0.234024E-03	-144.93	-0.11263E-04	0.31196E-05	0.54732E-08	0.94742E-04	-C.30433E-03
2	-C.89585E-01	-665.02	-0.15634E-02	0.59456	-0.81546E-02	0.64409E-02	0.28622	-137.89
3	1365.7	0.61935	0.24590E-02	0.21760E-02	-14.684	0.65773E-02	510.04	C.81061E-01
4	-0.67247	1.6533	-0.52833E-04	C.27632	0.85720E-01	-7932.7	-3.0339	12.596
5	56285.	-11.834	-0.18037E-01	0.26734	-1280.9	-0.16292	39158.	-9.1743
6	8.7428	12456.	-0.59710E-02	79.502	-0.13715	-0.17770	4.3503	4010.8
7	-0.17383E-02	-0.20957E-02	-708.80	-C.62935E-04	0.12952E-04	0.16760E-07	0.37724E-03	0.18000E-02
8	-0.49212E-01	-811.56	-0.34206E-03	-140.62	0.62452E-02	0.24019E-02	0.23122	-4281.8
9	1517.3	-0.52463E-01	0.31171E-03	C.65911E-02	-176.93	-0.32531E-02	5829.2	0.27422
10	-1.5709	-3.6138	0.31599E-04	0.14093	0.11388	-62785.	-4.3272	-5.0562
11	0.31547E-06	3.8601	0.51480E-01	0.18920	-4621.5	-0.21912	87731.	8.7651
12	-0.52567	80507.	0.12261E-01	1604.4	0.42394E-01	0.55054E-01	-1.4564	39793.
13	0.16105E-02	0.42859E-03	1932.0	-0.12655E-03	-0.51905E-05	0.10718E-06	0.25235E-04	0.49973E-02
14	0.88417E-01	1604.3	-0.32841E-04	166.30	-0.50911E-02	0.54898E-02	0.18424	4649.0
15	-4621.0	-0.24356	-0.25680E-03	0.48451E-02	244.81	0.31747E-02	-7557.4	-0.12748
16	-3.1555	3.0929	0.30525E-05	0.67313E-01	0.21982	0.10285E-06	-7.7748	1.5766
17	87722.	8.0988	-0.14499E-01	0.51261	-7557.2	0.55533E-01	0.39153E-06	19.321
18	7.1383	35793.	-0.20887E-01	4649.0	-0.24777	0.25682E-01	8.8463	0.22345E-06

HUB (ATTACHMENT POINT) STIFFNESS MATRIX

TABLE 10.11 (Cont'd)

AFY-K6HG

ROW COLUMN	1	2	3	4	5	6	7	8	9	10
1	-0.47199E 07	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	-0.26976E 06	0.0	0.0	0.0	-0.47477E 07	0.0	0.0	0.0	0.0
3	0.0	0.0	-0.26976E 06	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	-0.21426E 08	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	-0.47477E 07	0.0	0.55707E 08	0.0	0.0	0.0	0.0	0.0
6	0.0	0.47477E 07	0.0	0.0	0.0	0.55707E 08	0.0	0.0	0.0	0.0

AFY-K6HG

ROW COLUMN	11	12	13	14	15	16	17	18
1	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
2	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
3	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
4	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
5	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
6	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0

HUB (ATTACHMENT POINT) DAMPING MATRIX

AFY-CHG

ROW COLUMN	1	2	3	4	5	6	7	8	9	10
1	-1044.4	0.47952E-03	0.82768E-C4	0.26959E-06	-0.10136E-02	0.64918E-C2	-251.76	-0.25135E-03	-0.83102E-07	-0.26351E-06
2	0.10453E-02	-243.71	0.78193E-C2	0.62622E-02	0.57455E-02	-2781.0	-0.26243E-02	-22.565	-0.95205E-02	-0.11829E-01
3	0.29363E-03	0.10553E-02	-256.88	-0.15934E-03	5291.5	-1.0650	0.23477E-02	-0.11837E-01	-32.008	0.82276E-02
4	0.15670E-04	0.14108	-0.11473	-0.10976E 06	-0.78129	4.3272	-0.30205E-04	0.22309	0.47925	-41645.
5	0.22357E-01	0.27376	-4956.7	-0.12764	-4708.0	6.7361	0.41116E-C1	0.61146	-1273.2	0.15273
6	-0.41401E-01	4596.C	-0.38364	-1.5730	2.4169	16949.	0.10933	1375.7	0.15859	1.4415

AFY-CHG

ROW COLUMN	11	12	13	14	15	16	17	18
1	0.47742E-03	0.20114E-02	-1.078.3	0.20075E-03	-0.10881E-04	0.12942E-06	0.44675E-03	0.71015E-02
2	-0.49043E-01	-127.76	0.19382E-02	-25.079	0.70005E-02	0.75277E-02	-0.23924	-229.30
3	1738.1	-0.32341	-0.18905E-02	-0.39212E-02	-53.183	-0.68989E-02	1218.2	-0.22774
4	5.3989	-1.1325	0.58181E-04	-0.20270	-0.41941	-32131.	15.136	-9.1163
5	30201.	18.890	-0.38311E-C1	-0.41826	-5416.5	-0.80286E-01	27719.	-28.817
6	-0.81409	3764.5	-0.77492E-01	4895.7	-0.10613	-0.64192	3.6928	9340.3

HUB-VOCAL MATRIX EIGENVALUES

TABLE 10.11 (Cont'd)

OMEGA

EIG. VAL.-NO. 1 2 3 4 5 6 7 8 9 10
 0.45785E 00 0.47574E 00 0.14248E 01 0.35451E 01 0.42664E 01 0.56755E 01 0.57238E 01 0.61003E 01 0.11097E 02 0.14880E 02

OMEGA

EIG. VAL.-NO. 11 12 13 14 15 16 17 18
 0.17455E 02 0.21486E 02 0.26970E 02 0.30044E 02 0.30214E 02 0.50155E 02 0.87929E 02 0.13262E 03

HUB MODAL MATRIX

MODE-MAT

ROW COLUMN	1	2	3	4	5	6	7	8	9	10
1	C.59903E-14	0.67253E-12	C.10778E-13	0.30127E-11	0.15487E-11	C.19931E-08	0.19561E-09	0.75385E-09	C.36700E-08	0.22766E-06
2	C.13100E-07	0.79529E-01	0.68377E-06	0.35622E-06	0.31872E-03	0.39905	-0.16838E-01	0.73214E-05	-0.57577E-04	1.0000
3	0.78731E-01	0.24985E-09	C.18103E-05	C.35728	C.22814E-04	0.51476E-06	0.36507E-03	1.0000	-0.21158E-01	0.33728E-03
4	C.73242E-14	0.17984E-11	0.24450E-13	0.80109E-11	C.41353E-11	0.53096E-08	0.52312E-09	0.20999E-08	C.97019E-08	0.58863E-06
5	-0.40928E-08	0.46450	-0.14443E-05	0.17815E-05	0.94907E-03	1.0000	-0.41912E-01	0.33368E-04	C.10663E-04	0.25714
6	C.48272	-0.57904E-09	C.45339E-05	-1.0000	0.10095E-04	0.25144E-05	0.39470E-04	0.46009	1.0000	0.13211E-04
7	-0.60228E-11	0.62231E-11	C.10833E-11	C.14603E-06	0.13808E-05	0.98665E-08	0.10248E-08	C.13580E-07	0.35738E-07	0.74538E-06
8	0.22419E-09	1.0000	0.16927E-05	0.42843E-07	0.63031E-05	0.44949E-01	0.19427E-02	0.18458E-05	0.17975E-05	0.36568E-01
9	1.0000	-0.15845E-08	C.16151E-06	0.33455E-01	0.64533E-05	0.18043E-06	0.36841E-05	0.10629E-01	C.25647	-0.11811E-04
10	-0.14299E-09	0.58497E-08	0.35297	-0.84556E-08	1.0000	-0.75802E-05	0.56445	0.17177E-05	0.15723E-06	0.65415E-06
11	0.42890E-02	0.13034E-10	C.95088E-07	0.20029E-01	0.29770E-06	C.41225E-07	0.15273E-04	0.53264E-01	C.40338E-01	0.19277E-05
12	0.10359E-08	0.43184E-02	C.17705E-07	C.76307E-07	C.15550E-04	0.18209E-01	0.76150E-03	0.14195E-05	0.14090E-05	0.35291E-01
13	-0.57376E-11	0.11025E-05	C.84732	C.44138E-07	0.20594	0.15002E-04	1.0000	-0.31291E-05	C.14295E-06	0.99040E-07
14	C.88930E-02	0.16897E-10	0.21219E-08	0.45534E-02	0.67858E-06	0.13348E-06	0.27645E-04	0.96414E-01	-0.36457E-01	0.13793E-06
15	C.45598E-11	0.88683E-02	0.16248E-08	C.16400E-07	0.62534E-06	0.47381E-02	0.20028E-03	0.52543E-06	C.20626E-05	0.41237E-01
16	-0.52020E-11	0.12277E-09	1.0000	-0.16444E-07	0.55574	-0.10552E-04	0.468308	0.21075E-05	C.33688E-06	0.3577E-06
17	C.10071E-01	0.15653E-10	0.31676E-08	0.24929E-01	0.10259E-05	0.13182E-06	0.13709E-04	0.59078E-01	0.15115E-01	0.42810E-06
18	-0.20601E-09	0.10015E-01	C.10843E-06	0.74283E-07	0.26641E-04	0.26321E-01	0.11029E-02	0.65993E-06	0.13656E-05	0.29054E-01

MODE-MAT

ROW COLUMN	11	12	13	14	15	16	17	18
1	0.29052	-0.11455E-06	0.35793E-06	0.31106E-07	0.76811E-07	0.20639E-06	1.0000	0.51099
2	-0.10054E-05	0.55130E-03	-1.0000	0.55826E-03	C.14444	-0.12918	0.64147E-06	0.16263E-05
3	-0.79842E-06	-1.0000	-0.62928E-03	0.14478	-0.28023E-02	0.21804E-03	0.24692E-05	0.20252E-05
4	0.75441	-0.29272E-06	C.90107E-06	0.75861E-07	0.18560E-06	0.42914E-06	0.63942	-1.0000
5	-0.31666E-05	C.19436E-04	0.28484	-0.59356E-02	-1.0000	0.33635E-06	0.49223E-06	0.50188E-06
6	-0.51959E-08	C.43733E-01	0.40611E-03	-1.0000	0.13662E-01	0.95159E-04	0.39254E-06	0.50188E-06
7	1.0000	0.62784E-09	C.10920E-05	0.63232E-08	0.13291E-06	0.34023E-06	0.12220	C.75887E-01
8	C.13364E-06	0.23034E-04	0.28529E-01	0.41608E-03	0.69989E-01	0.16218	-0.14123E-07	0.15393E-07
9	C.11393E-07	0.35992E-01	0.37376E-04	C.12074	-0.16148E-02	0.24081E-04	0.27239E-07	0.40544E-07
10	-0.71994E-11	0.81556E-07	0.23005E-06	0.17776E-06	0.74551E-06	0.73276E-06	0.23402E-10	0.33213E-10
11	C.42928E-08	0.40175E-02	0.20321E-05	0.65477E-03	0.61358E-05	0.17536E-05	0.50590E-08	0.69192E-08
12	0.13420E-06	C.44093E-06	0.50368E-01	0.24390E-03	-0.39556E-01	0.59215E-02	0.11772E-07	0.13920E-07
13	C.36116E-11	0.17531E-06	0.49821E-06	0.14681E-06	0.18168E-06	0.62761E-06	0.80440E-11	0.11418E-10
14	C.14289E-06	0.33697E-02	C.97105E-06	0.32799E-02	0.42227E-04	0.11757E-05	0.13464E-07	0.18842E-07
15	-0.12483E-08	0.28314E-04	C.24847E-01	-0.38950E-03	0.65735E-01	0.16219	-0.92440E-07	0.11707E-06
16	-0.10671E-11	0.13658E-06	0.55190E-06	0.39928E-06	C.19735E-07	0.43330E-06	0.88970E-12	0.12278E-11
17	-0.26495E-08	0.24352E-02	C.41566E-05	0.8348E-02	0.11295E-03	0.22098E-05	0.3348E-08	0.45693E-08
18	0.70863E-07	0.15854E-04	0.11057E-01	0.15459E-03	0.25926E-01	0.43708E-01	0.15922E-07	0.23235E-07

11.0 REFERENCES

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5. Revision 1 to Reference 4, July 15, 1971.
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8. "Stability Criteria for a Rotating Space Station with a Non-rotating Hub," Austin, F.; AIAA Journal, Vol. 6, No. 11, November 1968.
9. "Use of Floquet Theory to Determine the Stability of Linear Differential Equations with Periodic Coefficients," Austin, F.; Grumman Aerospace Corp., Structural Mechanics IOM 70.38, May 1970.
10. "Optimum Wobble Damping of Rotating Satellites Using a Control-Moment Gyroscope," Austin, F.; Berman, H.; Grumman Report ADRO6-05-71.1, June 1971.

where

$$k_1 = \frac{AE}{L}$$

$$k_4 = \frac{GJ}{L}$$

$$k_2 = \frac{12EI}{L^3}$$

$$k_5 = \frac{4EI}{L}$$

$$k_3 = \frac{6EI}{L^2}$$

$$k_6 = \frac{2EI}{L}$$

Incremental Beam Element Stiffness Matrix

The following incremental stiffness matrix will be added to $[K^*]$ to account for the stiffening effect due to the centrifugal forces

P** = axial tensile load applied to the beam; the direction of this load does not change as beam bends

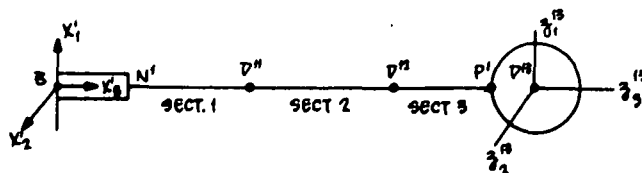


$$[K^{**}] = \frac{P^{**}}{10L}$$

12									
	12								
								(SYMMETRIC)	
		-L	$\frac{4L^2}{3}$						
L			$\frac{4L^2}{3}$						
				-L	12				
-12									
	-12	L				12			
		-L	$\frac{4L^2}{3}$			L	$\frac{4L^2}{3}$		
L			$\frac{4L^2}{3}$				$\frac{4L^2}{3}$		

Stiffness Of A Rotor Arm

Next the element stiffness matrices will be assembled to obtain the stiffness matrix for a rotor arm.



$[K1^*]$, $[K2^*]$ and $[K3^*]$ denote basic stiffness for elements 1, 2 and 3, respectively.

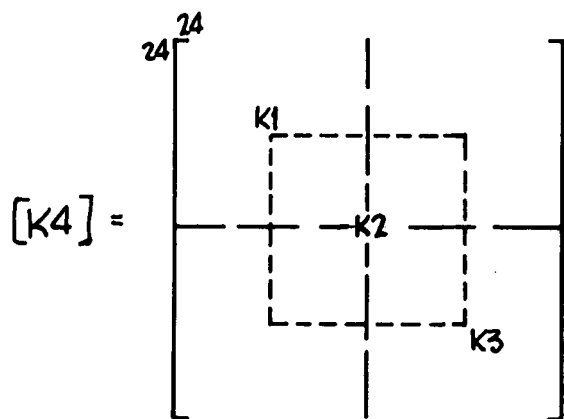
If P_1 , P_2 , P_3 are centrifugal forces due to M_{D11} , M_{D12} and M_{D13} , respectively, then

$$P_1 = m_{D11} \Omega^2 [0,0,1] \{x_{D11}\}, P_2 = m_{D12} \Omega^2 [0,0,1] \{x_{D12}\}, P_3 = m_{D13} \Omega^2 [0,0,1] \{x_{D13}\}$$

For the linear elastic analysis it is sufficiently accurate to express the axial tensile loads in Section 1, 2 and 3 as $(P_1 + P_2 + P_3)$, $(P_2 + P_3)$ and (P_3) , respectively. These loads can now be used to form $[K1^{**}]$, $[K2^{**}]$ and $[K3^{**}]$ which are the incremental stiffness of elements 1, 2 and 3 respectively. Next the total element stiffness matrix is obtained by combining the basic and incremental stiffnesses:

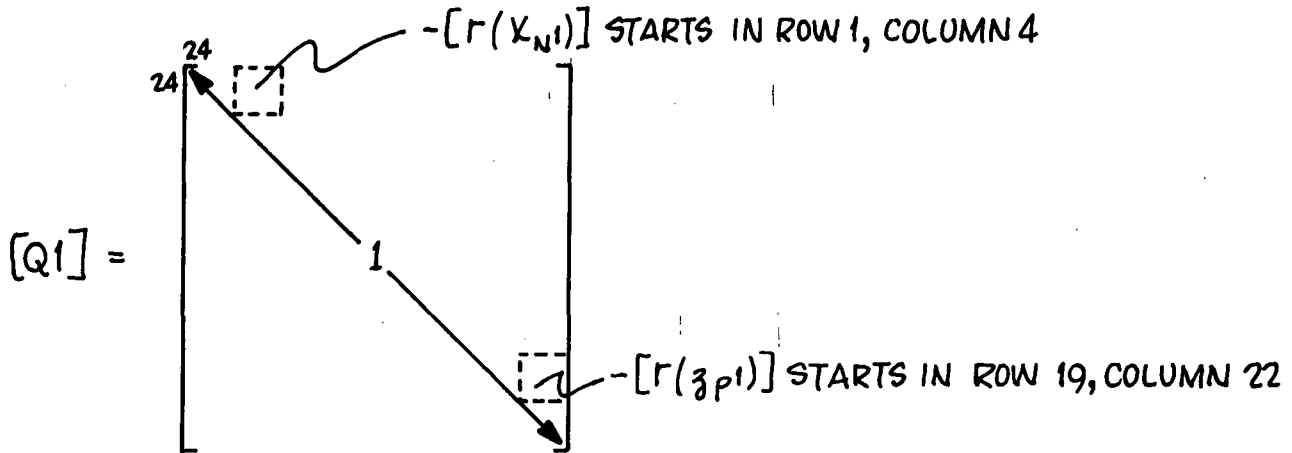
$$\begin{aligned} [K1] &= [K1^*] + [K1^{**}] \\ [K2] &= [K2^*] + [K2^{**}] \\ [K3] &= [K3^*] + [K3^{**}] \end{aligned}$$

The above matrices are now stacked to obtain an assembled stiffness matrix for the three beam elements.



$[K1]$ STARTS IN ROW 1, COLUMN 1
 $[K2]$ STARTS IN ROW 7, COLUMN 7
 $[K3]$ STARTS IN ROW 13, COLUMN 13

Matrix $[K4]$ operates on displacements at points N^1 , D^{11} , D^{12} and P^1 . Matrix $[Q1]$ is formed to transform to displacements at B , D^{11} , D^{12} and D^{13} .



The transformed stiffness matrix is $[K5]$:

$$[K5] = [Q1]^T [K4] [Q1]$$

Adjustments To $[K5]$:

The previous transformation does not include all linear terms involving the axial tensile loads; thus, the following correction terms are provided:

$$k_7 = (P_1 + P_2 + P_3) [0, 0, 1] \{x_{N1}\}$$

$$k_8 = -(P_3) [0, 0, 1] \{z_{P1}\}$$

k_7 is added to the (4, 4) and (5, 5) elements of $[K5]$; also, k_8 is added to the (22, 22) and (23, 23) elements of $[K5]$. The result is denoted as $[K5A]$; this matrix will lead to the stiffness matrix supplied to the time history program.

Addition of External Loads Simulating Centrifugal Effects For Eigenvalue Problem

In order to properly form the stiffness matrix for the eigenvalue problem centrifugal loads P_1 , P_2 and P_3 are now added to the vehicle idealization.

As was mentioned previously the incremental stiffness matrix $[K^{**}]$ is based on the assumption that the axial load does not change direction as the beam bends. Since centrifugal forces pass through the axis of rotation, they retain their direction only when a rotor mass moves out of the spin plane; the following correction terms are needed for rotor motion in the spin plane:

$$k_9 = P_1 / [0, 0, 1] \{x_{D11}\}, \quad k_{10} = P_2 / [0, 0, 1] \{x_{D12}\}, \quad k_{11} = P_3 / [0, 0, 1] \{x_{D13}\}$$

k_9 , k_{10} and k_{11} are added to [K5A] as follows:

k_9 to the (2, 8) and (8, 2) elements

k_{10} to the (2, 14) and (14, 2) elements

k_{11} to the (2, 20) and (20, 2) elements

$-(k_9 + k_{10} + k_{11})$ to the (2, 2) element

$-k_9$ to the (8, 8) element

$-k_{10}$ to the (14, 14) element

$-k_{11}$ to the (20, 20) element

The result is denoted as [K5B]; this matrix will lead to the stiffness matrix used in the eigenvalue problem.

The static axial displacement due to centrifugal loads is not included in the eigenvalue problem. These static displacements are as follows:

$$\text{POINT } D^I : \frac{(P_1 + P_2 + P_3) L_1}{AE} + \frac{(P_2 + P_3) L_2}{AE} + \frac{P_3 L_3}{AE}$$

$$\text{POINT } D^{II} : \frac{(P_2 + P_3) L_3}{AE} + \frac{P_3 L_3}{AE}$$

$$\text{POINT } D^{III} : \frac{P_3 L_3}{AE}$$

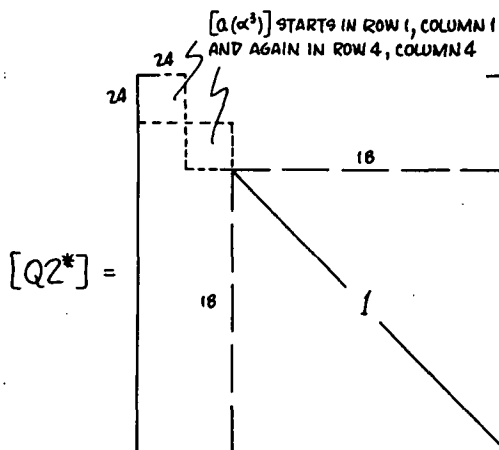
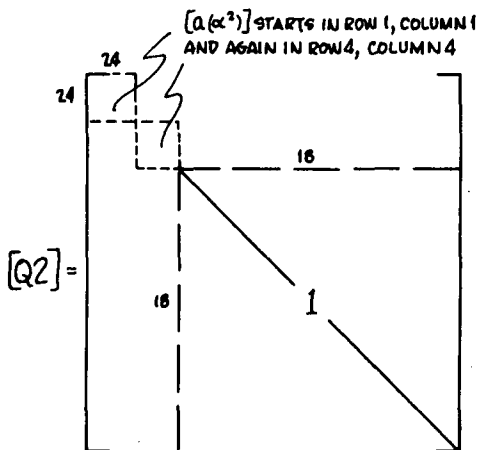
Stiffness of Entire Rotor

As all arms in a rotor are identical, matrices [K5A] and [K5B] can represent the stiffness of any arm in local (not global) coordinates; i.e., in coordinates aligned with that arm. In order to assemble the rotor, the displacement at point B (which is common to all arms) must be transformed to a common coordinate system; let the displacements at B be measured in \tilde{x}^1 coordinates. The stiffness matrix of the first arm is already in \tilde{x}^1 coordinates and, therefore,

requires no transformation. The second and third arm stiffness matrices will be transformed using the following matrices:

Second arm:

Third arm (if present):



$$[K6A] = [Q2]^T [K5A][Q2]$$

$$[K6A^*] = [Q2^*]^T [K5A][Q2^*]$$

$$[K6B] = [Q2]^T [K5B][Q2]$$

$$[K6B^*] = [Q2^*]^T [K5B][Q2^*]$$

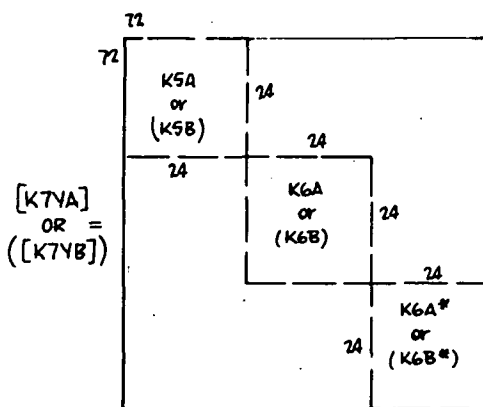
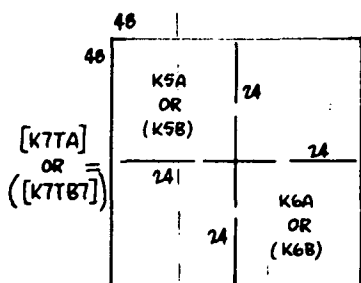
Assemble Rotor

Next the stiffness matrix of the entire rotor is formed by noting that the displacements at point B are common to all arms. This is accomplished by carrying out the following steps:

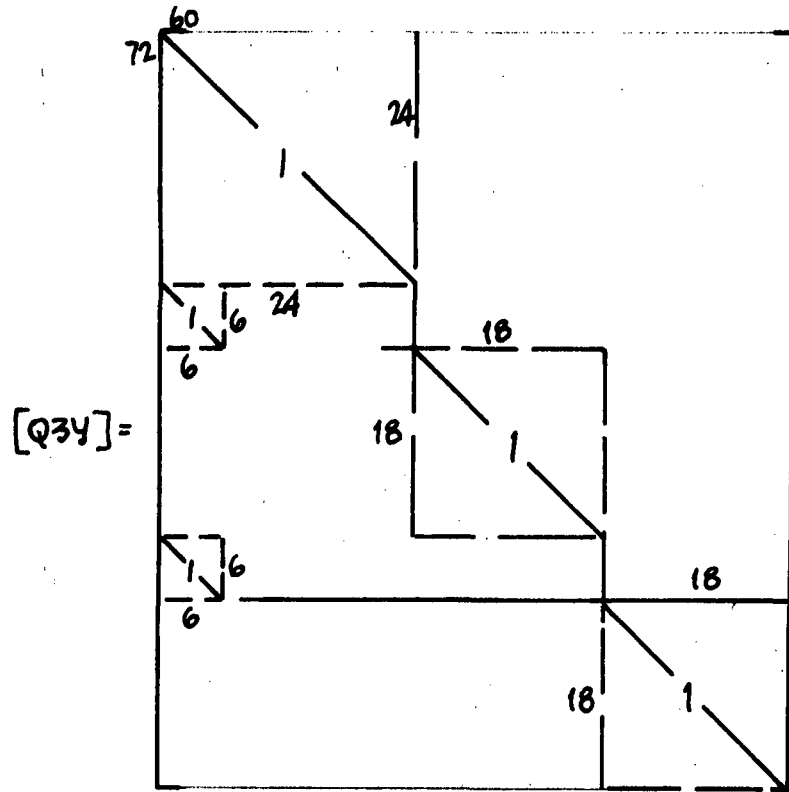
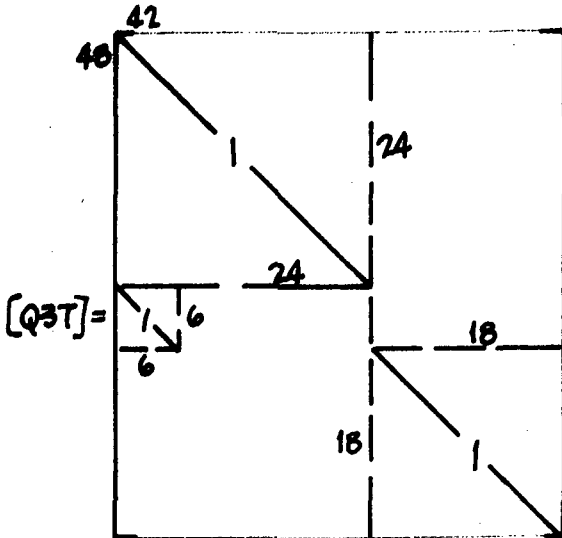
The previously obtained arm stiffness matrices are first stacked.

T Rotor

Y Rotor



Next the following matrices are formed to superimpose the displacements at point B.



The matrices are now transformed as follows:

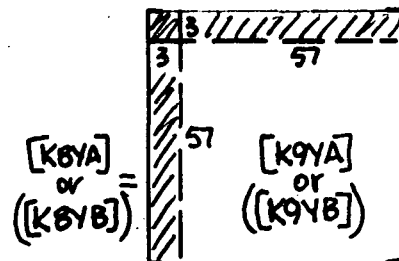
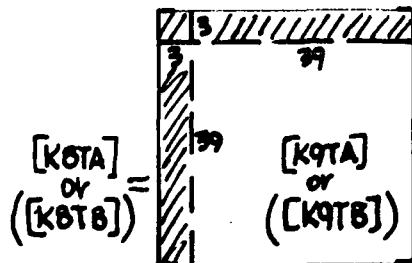
$$[K8TA] = [Q3T]^T [K7TA] [Q3T]$$

$$[K8TB] = [Q3T]^T [K7TB] [Q3T]$$

$$[K8YA] = [Q3Y]^T [K7YA] [Q3Y]$$

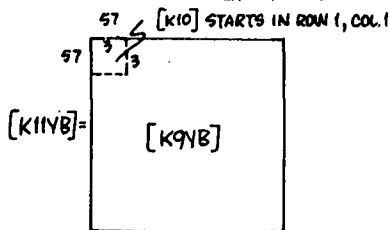
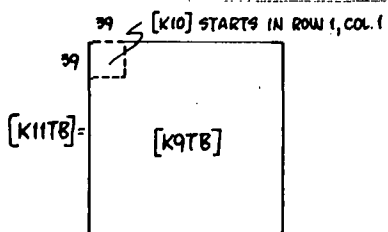
$$[K8YB] = [Q3Y]^T [K7YB] [Q3Y]$$

Next the rotor is pinned at point B by deleting the first three rows and columns.



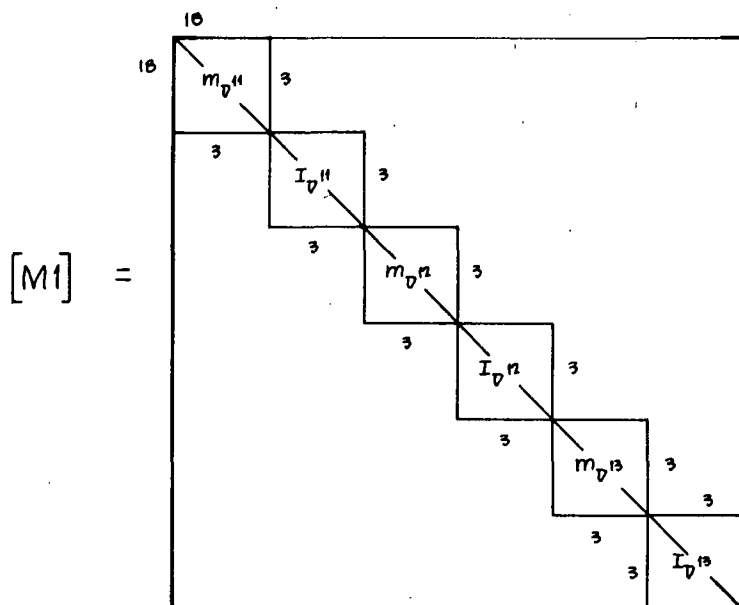
Except for a transformation to relative coordinates, stiffness matrices [K9 TA] and [K9 YA] are the rotor stiffness matrices for the time-history simulation. Whereas the bearing spring already appears in the time-history equations of Section 6.3, it must now be incorporated into the eigenvalue problem. This is accomplished as follows:

$$[K10] = K_{B41} \begin{bmatrix} 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

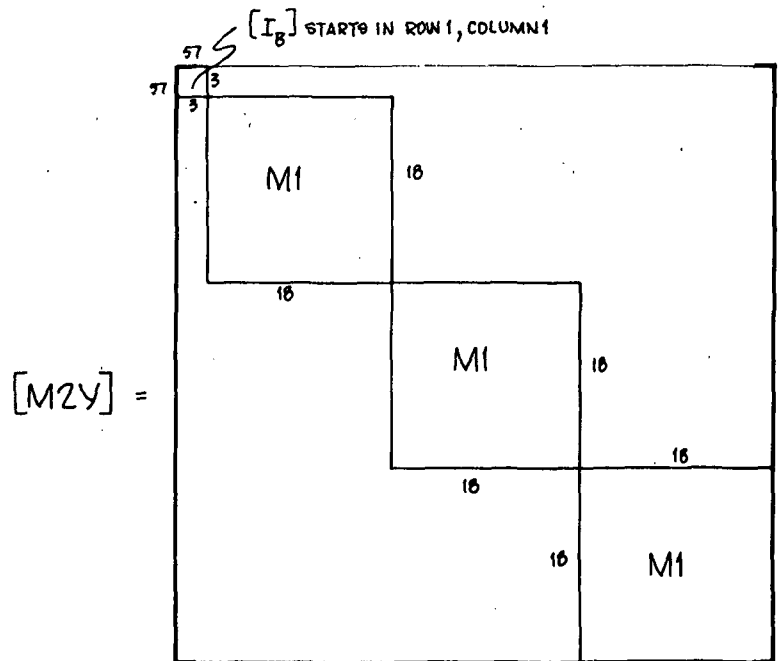
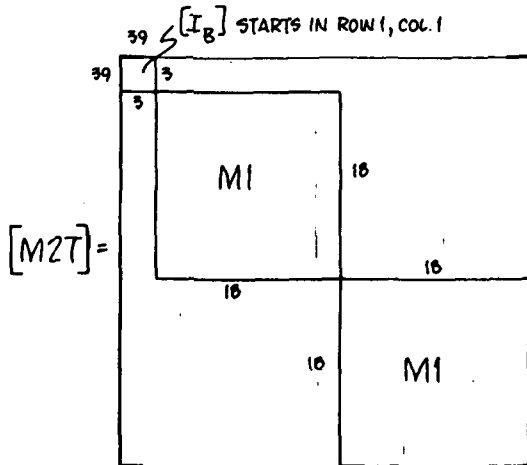


Mass Matrix

The mass matrix for one rotor arm is as follows:



In order to obtain the mass matrix for the entire rotor the matrices for each arm and for the rotor ring are stacked as follows:



Rotor Eigenvalue Problem

The eigenvalue problem for each simulation vehicle is formulated in this section.

The equations of motion follow:

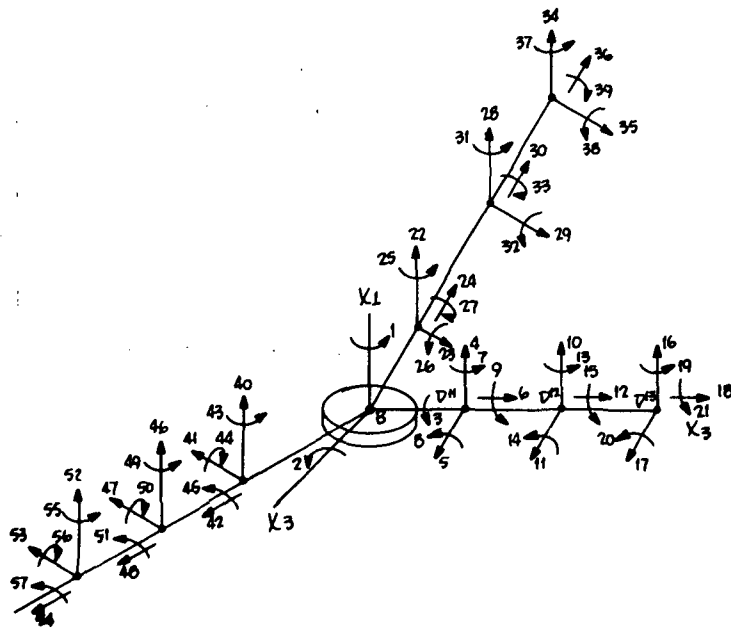
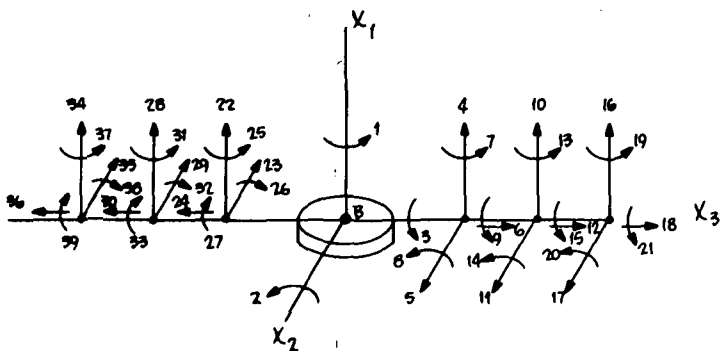
T Rotor

$$[M_{2T}]\{\ddot{q}_T\} + [K_{11TB}]\{q_T\} = 0$$

Y Rotor

$$[M_{2Y}]\{\ddot{q}_Y\} + [K_{11YB}]\{q_Y\} = 0$$

The displacements $\{q_T\}$ and $\{q_Y\}$ are shown below:



T Rotor

Symmetric motion (with respect to the X_1-X_2 plane) is characterized by:

ROTOR RING

$\theta_1 = 0$
 $\theta_2 = 0$
 $\theta_3 = \theta_5$

ARMS

$\theta_4 = \theta_{22}$
 $\theta_5 = -\theta_{23}$
 $\theta_6 = \theta_{24}$
 $\theta_7 = -\theta_{25}$
 $\theta_8 = \theta_{26}$
 $\theta_9 = -\theta_{27}$
 $\theta_{10} = \theta_{28}$
 $\theta_{11} = -\theta_{29}$
 $\theta_{12} = \theta_{30}$
 $\theta_{13} = -\theta_{31}$
 $\theta_{14} = \theta_{32}$
 $\theta_{15} = -\theta_{33}$
 $\theta_{16} = \theta_{34}$
 $\theta_{17} = -\theta_{35}$
 $\theta_{18} = \theta_{36}$
 $\theta_{19} = -\theta_{37}$
 $\theta_{20} = \theta_{38}$
 $\theta_{21} = -\theta_{39}$

19 DEGREES OF FREEDOM

Y Rotor

Symmetric motion (with respect to the X_1-X_3 plane) is characterized by:

ROTOR RING

$\theta_1 = 0$
 $\theta_2 = \theta_3$
 $\theta_4 = 0$

FIRST ARM

$\theta_4 = \theta_4$
 $\theta_5 = 0$
 $\theta_6 = \theta_6$
 $\theta_7 = 0$
 $\theta_8 = \theta_8$
 $\theta_9 = 0$
 $\theta_{10} = \theta_{10}$
 $\theta_{11} = 0$
 $\theta_{12} = \theta_{12}$
 $\theta_{13} = 0$
 $\theta_{14} = \theta_{14}$
 $\theta_{15} = 0$
 $\theta_{16} = \theta_{16}$
 $\theta_{17} = 0$
 $\theta_{18} = \theta_{18}$
 $\theta_{19} = 0$
 $\theta_{20} = \theta_{20}$
 $\theta_{21} = 0$

SECOND AND

THIRD ARMS

$\theta_{22} = \theta_{40}$
 $\theta_{23} = -\theta_{41}$
 $\theta_{24} = \theta_{42}$
 $\theta_{25} = -\theta_{43}$
 $\theta_{26} = \theta_{44}$
 $\theta_{27} = -\theta_{45}$
 $\theta_{28} = \theta_{46}$
 $\theta_{29} = -\theta_{47}$
 $\theta_{30} = \theta_{48}$
 $\theta_{31} = -\theta_{49}$
 $\theta_{32} = \theta_{50}$
 $\theta_{33} = -\theta_{51}$
 $\theta_{34} = \theta_{52}$
 $\theta_{35} = -\theta_{53}$
 $\theta_{36} = \theta_{54}$
 $\theta_{37} = -\theta_{55}$
 $\theta_{38} = \theta_{56}$
 $\theta_{39} = -\theta_{57}$

25 DEGREES OF FREEDOM

Antisymmetric motion (with respect to the X_1-X_2 plane) is characterized by:

ROTOR RING

$\theta_1 = \theta_1$
 $\theta_2 = \theta_2$
 $\theta_3 = 0$

ARMS

$\theta_4 = -\theta_{22}$	$\theta_{10} = -\theta_{28}$	$\theta_{16} = -\theta_{34}$
$\theta_5 = \theta_{23}$	$\theta_{11} = \theta_{29}$	$\theta_{17} = \theta_{35}$
$\theta_6 = -\theta_{24}$	$\theta_{12} = -\theta_{30}$	$\theta_{18} = -\theta_{36}$
$\theta_7 = \theta_{25}$	$\theta_{13} = \theta_{31}$	$\theta_{19} = \theta_{37}$
$\theta_8 = -\theta_{26}$	$\theta_{14} = -\theta_{32}$	$\theta_{20} = -\theta_{38}$
$\theta_9 = \theta_{27}$	$\theta_{15} = \theta_{33}$	$\theta_{21} = \theta_{39}$

20 DEGREES OF FREEDOM

Antisymmetric motion (with respect to the X_1-X_3 plane) is characterized by:

ROTOR RING

$\theta_1 = \theta_1$
 $\theta_2 = 0$
 $\theta_3 = \theta_3$

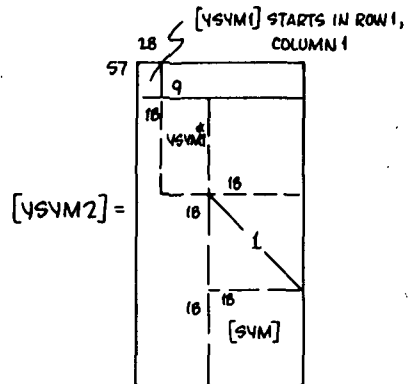
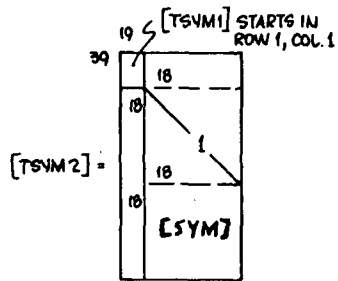
ARM 1

$\theta_4 = 0$	$\theta_{10} = 0$	$\theta_{16} = 0$
$\theta_5 = \theta_5$	$\theta_{11} = \theta_{11}$	$\theta_{17} = \theta_{17}$
$\theta_6 = 0$	$\theta_{12} = 0$	$\theta_{18} = 0$
$\theta_7 = \theta_7$	$\theta_{13} = \theta_{13}$	$\theta_{19} = \theta_{19}$
$\theta_8 = 0$	$\theta_{14} = 0$	$\theta_{20} = 0$
$\theta_9 = \theta_9$	$\theta_{15} = \theta_{15}$	$\theta_{21} = \theta_{21}$

ARMS 2 & 3

$\theta_{22} = -\theta_{40}$	$\theta_{28} = -\theta_{46}$	$\theta_{34} = -\theta_{52}$
$\theta_{23} = \theta_{41}$	$\theta_{29} = \theta_{47}$	$\theta_{35} = \theta_{53}$
$\theta_{24} = -\theta_{42}$	$\theta_{30} = -\theta_{48}$	$\theta_{36} = -\theta_{54}$
$\theta_{25} = \theta_{43}$	$\theta_{31} = \theta_{49}$	$\theta_{37} = \theta_{55}$
$\theta_{26} = -\theta_{44}$	$\theta_{32} = -\theta_{50}$	$\theta_{38} = -\theta_{56}$
$\theta_{27} = \theta_{45}$	$\theta_{33} = \theta_{51}$	$\theta_{39} = \theta_{57}$

29 DEGREES OF FREEDOM



$$[TANT1] = \begin{bmatrix} 1 & 0 \\ 0 & 1 \\ 0 & 0 \end{bmatrix}$$

$$[YANT1] = \begin{bmatrix} 1 & 0 \\ 0 & 0 \\ 0 & 1 \end{bmatrix}$$

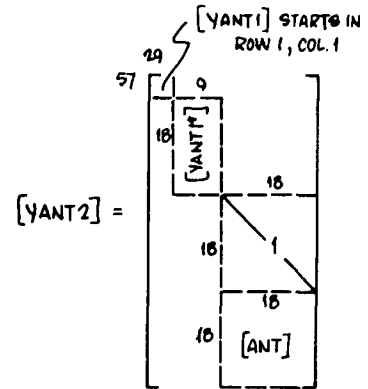
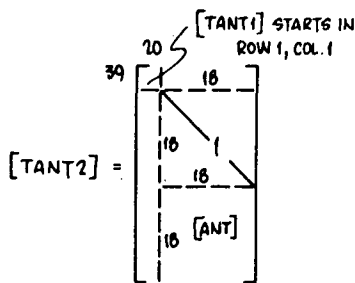
9

18

[YANT1*] =

0									
1									
	0								
		0							
			0						
				0					
					0				
						0			
							0		
								0	
									0

$$[ANT] = -[SYM]$$



The symmetric and antisymmetric eigenvalue problem are now set up as indicated below:

$$\begin{array}{ll}
 {}^{19} [M_{TSYM}] = [T_{SYM2}]^T [M_{2T}] [T_{SYM2}] & \text{--- SYMMETRIC MASS ---} \\
 {}^{19} [K_{TSYM}] = [T_{SYM2}]^T [K_{11TB}] [T_{SYM2}] & \text{--- SYMMETRIC STIFFNESS ---} \\
 [M_{TSYM}] \{\ddot{q}_T\}_S + [K_{TSYM}] \{q_T\}_S = 0 & \text{--- BASIC SYMMETRIC EQUATION ---} \\
 {}^{20} [M_{TANT}] = [TANT2]^T [M_{2T}] [TANT2] & \text{--- ANTISYMMETRIC MASS ---} \\
 {}^{20} [K_{TANT}] = [TANT2]^T [K_{11TB}] [TANT2] & \text{--- ANTISYMMETRIC STIFFNESS ---} \\
 [M_{TANT}] \{\ddot{q}_T\}_A + [K_{TANT}] \{q_T\}_A = 0 & \text{--- BASIC ANTISYMMETRIC EQUATION ---} \\
 {}^{26} [M_{YSYM}] = [Y_{SYM2}]^T [M_{2Y}] [Y_{SYM2}] & \\
 {}^{26} [K_{YSYM}] = [Y_{SYM2}]^T [K_{11YB}] [Y_{SYM2}] & \\
 [M_{YSYM}] \{\ddot{q}_Y\}_S + [K_{YSYM}] \{q_Y\}_S = 0 & \\
 {}^{29} [M_{YANT}] = [YANT2]^T [M_{2Y}] [YANT2] & \\
 {}^{29} [K_{YANT}] = [YANT2]^T [K_{11YB}] [YANT2] & \\
 [M_{YANT}] \{\ddot{q}_Y\}_A + [K_{YANT}] \{q_Y\}_A = 0 &
 \end{array}$$

In order to obtain the mode shapes $[\phi]$ and frequencies ω , the above eigenvalue problems must be solved. The results are denoted as follows:

$$\begin{array}{ll}
 \text{SYMMETRIC MODES: } {}^{19} [\phi_{TSYM1}] & \text{SYMMETRIC MODES: } {}^{26} [\phi_{YSYM1}] \\
 \text{ANTISYMMETRIC MODES: } {}^{20} [\phi_{TANT1}] & \text{ANTISYMMETRIC MODES: } {}^{29} [\phi_{YANT1}]
 \end{array}$$

Next the modal vectors are expanded to full size

$$\begin{array}{ll}
 {}^{39} [\phi_{TSYM2}] = {}^{39} [T_{SYM2}] {}^{19} [\phi_{TSYM1}] & {}^{26} [\phi_{YSYM2}] = [Y_{SYM2}] [{}^{26} \phi_{YSYM1}] \\
 {}^{39} [\phi_{TANT2}] = {}^{39} [TANT2] {}^{20} [\phi_{TANT1}] & {}^{57} [\phi_{YANT2}] = [YANT2] [{}^{29} \phi_{YANT1}]
 \end{array}$$

The modal matrices are now partitioned into modes to be retained and modes to be ignored in the modal reduction procedure.

$$\begin{array}{l} \begin{array}{cc} \text{(retain)} & \text{(ignore)} \\ [\phi_{TSYM2}] = [\phi_{TSYM3}; \phi_{TSYM4}] \end{array} & \begin{array}{cc} \text{(retain)} & \text{(ignore)} \\ [\phi_{YSYM2}] = [\phi_{YSYM3}; \phi_{YSYM4}] \end{array} \\ \begin{array}{cc} \text{(retain)} & \text{(ignore)} \\ [\phi_{TANT2}] = [\phi_{TANT3}; \phi_{TANT4}] \end{array} & \begin{array}{cc} \text{(retain)} & \text{(ignore)} \\ [\phi_{YANT2}] = [\phi_{YANT3}; \phi_{YANT4}] \end{array} \end{array}$$

The symmetric and antisymmetric modal matrices of the n_t modes to be retained are:

The symmetric and antisymmetric modal matrices of the n_y modes to be returned are:

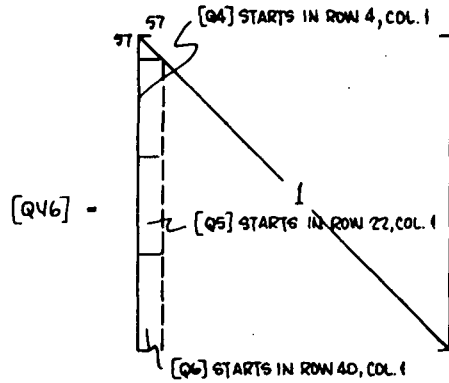
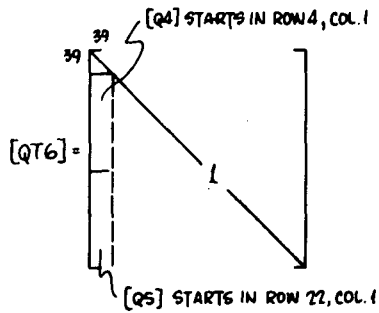
$$\begin{array}{l} \begin{array}{c} n_t \\ 39 \end{array} [\phi_T] = [\phi_{TSYM3}; \phi_{TANT3}] & \begin{array}{c} n_y \\ 57 \end{array} [\phi_Y] = [\phi_{YSYM3}; \phi_{YANT3}] \end{array}$$

The above mode shapes are expressed in local absolute coordinates, whereas the equations of motion in the time history program are in local relative coordinates. The mode shapes can be converted into relative coordinates by use of transformation matrices [QT6] and [QY6] defined as follows:

$$[Q4] = \begin{array}{c} 3 \\ 16 \\ \begin{array}{|c|} \hline [\phi(x_p^1)] \\ \hline -1 \\ \hline -1 \\ \hline -1 \\ \hline [\phi(x_p^2)] \\ \hline -1 \\ \hline -1 \\ \hline -1 \\ \hline [\phi(x_p^3)] \\ \hline -1 \\ \hline -1 \\ \hline -1 \\ \hline \end{array} \end{array}$$

$$[Q5] = [Q4][a(\alpha^2)]$$

$$[Q5^*] = [Q4][a(\alpha^3)]$$

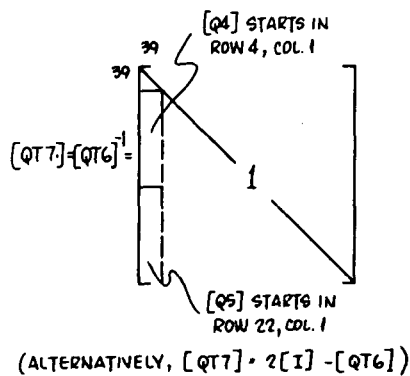


Next the modal matrices are transformed to relative coordinates:

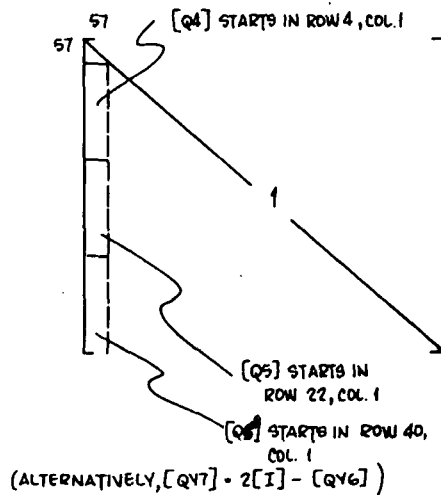
$${}^{39}\{\phi TR\} = [QT6][\phi T]$$

$${}^{57}\{\phi VR\} = [QY6][\phi Y]$$

The stiffness matrices needed for the time history program must also be converted into relative coordinates.



$$[KTA] = [K9TA][QT7]$$



$$[KYA] = [K9YA][QY7]$$

Rotor Damping

It was decided to use modal damping to represent energy dissipation in the structure. Since inversion of the square modal matrix is used in manipulations below, all modes are now assembled.

$$^{39} [\Phi TALL] = [\Phi TSYM2'; \Phi TANT2] \quad ^{57} [\Phi YALL] = [\Phi YSYM2'; \Phi YANT2]$$

The following frequency matrices will also be required.

$$[WT] = \begin{array}{c|c} \begin{array}{c} \text{WTSYM} \\ 19 \end{array} & \begin{array}{c} 19 \\ \text{---} \\ 20 \end{array} \\ \hline \begin{array}{c} 19 \\ \text{---} \\ 20 \end{array} & \begin{array}{c} \text{WTANT} \end{array} \end{array}$$

$$[WY] = \begin{array}{c|c} \begin{array}{c} \text{WYSYM} \\ 26 \end{array} & \begin{array}{c} 26 \\ \text{---} \\ 29 \end{array} \\ \hline \begin{array}{c} 26 \\ \text{---} \\ 29 \end{array} & \begin{array}{c} \text{WYANT} \end{array} \end{array}$$

The following physical damping matrix is based upon a fraction, γ , of critical modal damping:

$$[CT] = 2\gamma [M2T][\Phi TALL][\omega T][\Phi TALL]^{-1} \quad | \quad [CY] = 2\gamma [M2Y][\Phi YALL][\omega Y][\Phi YALL]^{-1}$$

Recalling that the bearing springs were incorporated into the stiffness matrices used for the modal analyses, it follows that the physical damping effects, as given by $[CT]$ and $[CY]$, will be transmitted to the hub. This unrealistic effect is deleted as follows:

$$[CT^*] = [QT7]^T [CT] [QT7]$$

The first 3 rows and columns of $[CT^*]$ are deleted, and the result is called $[CT^{**}]$. The final damping matrix is:

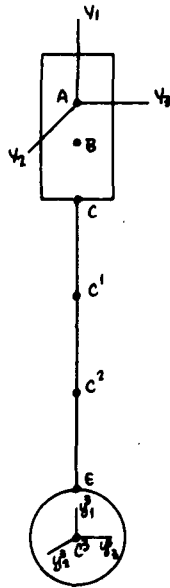
$$[CTA] = [QT7]^{-T} [CT^{**}] = [QT6] [CT^{**}]$$

$$[CY^*] = [QY7]^T [CY] [QY7]$$

The first 3 rows and columns of $[CY^*]$ are deleted and the result is called $[CY^{**}]$. The final damping matrix is:

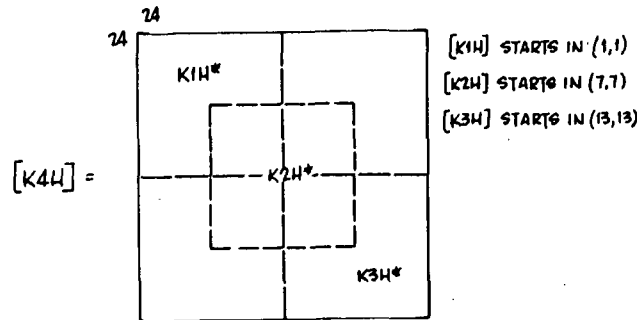
$$[CYA] = [QY7]^{-T} [CY^{**}] = [QY6] [CY^{**}]$$

II. Hub

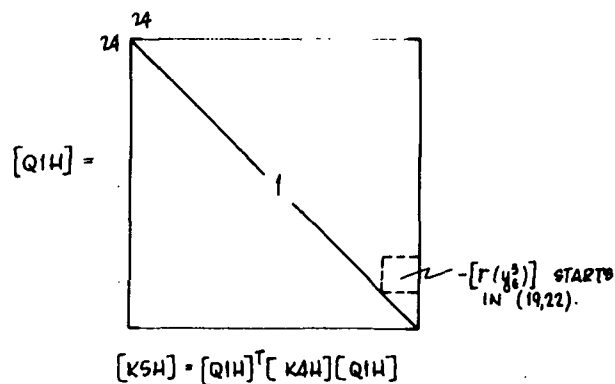


The basic beam stiffness matrix, $[K^*]$, presented in the rotor section is used to compute $[K1H^*]$, $[K2H^*]$ and $[K3H^*]$ for hub elements 1, 2 and 3, respectively. Note that the coordinate systems are not the same; that is, the Y_1 axis is the torsional axis for the hub, whereas the X_3 axis was the torsional axis in the rotor arm.

Next the stiffness elements are stacked to form the stiffness matrix of the total hub.



Matrix $[K4H]$ operates on displacements at points G, C^1 , C^2 and E. The transformation matrix $[Q1H]$ is now formed to convert to displacements of G, C^1 , C^2 and C^3 ; call result $[K5H]$.



To cantilever the hub at point G, the first six columns of $[K5H]$ are deleted below. The result is called $[K6H]$.

$$[K5H] = \begin{array}{c} 24 \\ \left[\begin{array}{c|c} & \\ \hline & 18 \\ & 24[K6H] \end{array} \right] \end{array}$$

$[K6H]$ is partitioned into $[K6HG]$ and $[K7H]$ as shown below:

$$[K6H] = \begin{array}{c} 6 \\ \left[\begin{array}{c} \text{hatched } [K6HG] \\ \hline 18 \\ [K7H] \end{array} \right] \end{array}$$

$[K7H]$ is the stiffness matrix for the flexible section of hub. $[K6HG]$ can be used to obtain the elastic loads transmitted to the rigid section of hub at point G.

The hub mass matrix is now stacked as follows:

$$[MH] = \begin{array}{c} 18 \\ \left[\begin{array}{c} MC^1 \\ \hline IC^1 \\ \hline MC^2 \\ \hline IC^2 \\ \hline MC^3 \\ \hline IC^3 \end{array} \right] \end{array}$$

$[MH]$ and $[K7H]$ are used to formulate the eigenvalue problem, i.e.,

$$[MH] \{\ddot{q}_H\} + [K7H] \{q_H\} = 0$$

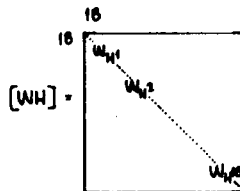
The solution of the above problem provides the modal matrix $[\phi_H]$ and frequencies, ω .

$[\phi_H^*]$ is partitioned into n_h modes to be retained for modal reduction and modes to be ignored.

$$[\phi_H] = \begin{matrix} \text{(retain)} & \text{(ignore)} \\ \left[\phi_H \mid \phi_H^{**} \right] \end{matrix}$$

Hub Damping

As was the case for the rotor, modal damping is used in the hub. The complete modal matrix $[\phi_H^*]$ as well as the following frequency matrix are required to obtain the hub damping matrix:



Finally the physical damping matrix is computed based upon a fraction, δ , of critical modal damping. The damping matrix for the flexible section of hub is:

$${}^{18} [CH] = 2\delta [MH] [\phi_H^*] [W_H] [\phi_H^*]^{-1}$$

and the damping forces transmitted to the rigid section of hub can be computed using $[CHG]$.

$${}^6 [CHG] = [K6HG] [K7H]^{-1} [CH]$$

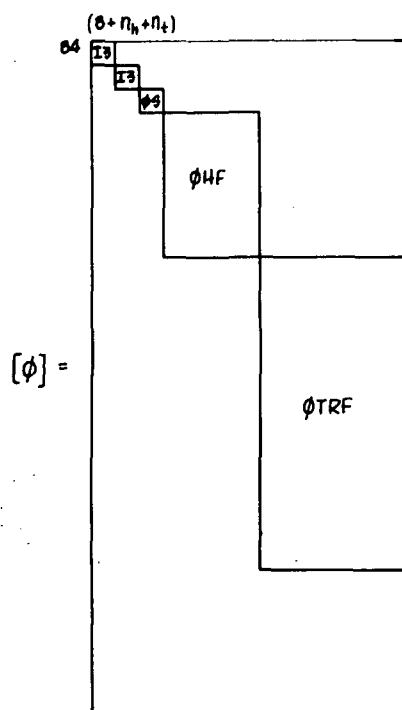
Assemble Total Reduction Matrix For Total Vehicle

In the time-history simulation, the hybrid coordinates used for numerical integration are the physical coordinates for the rigid section of hub $\{Z_A\}$ and $\{W^Y\}$, the two bending rotation of the shuttle ψ_1 and ψ_2 , the n_h hub modal coordinates, and the n_T or n_Y rotor modal coordinates. In order to rigidize the shuttle torsional coordinate ψ_3 , the following constraint matrix is formed:

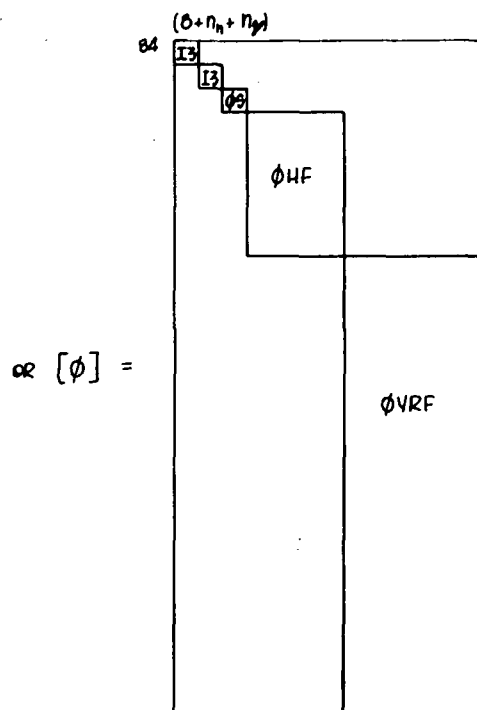
$$[\phi S] = \begin{bmatrix} 1 & 0 \\ 0 & 1 \\ 0 & 0 \end{bmatrix}$$

Then, the total matrix used to reduce the number of coordinates for numerical integration is:

For T Configuration



For Y Configuration



APPENDIX B

FREQUENCY SHIFT TECHNIQUE

The fundamental vibration equation may be written as

$$\omega_i^2 [M] \{\phi_i\} = [K] \{\phi_i\} \quad (\text{B-1})$$

where ω_i is the frequency and $\{\phi_i\}$ is the associated mode shape. Since the eigenvalue algorithm used in this study cannot treat a zero frequency, this frequency was removed from the equations in the following manner. Let $\bar{\omega}_i$ be a new frequency which is related to ω_i by the small constant shift frequency ω_0 by

$$\bar{\omega}_i^2 = \omega_i^2 + \omega_0^2 \quad (\text{B-2})$$

The fundamental equation (B-1) may then be written as

$$\bar{\omega}_i^2 [M] \{\phi_i\} = ([K] + \omega_0^2 [M]) \{\phi_i\} \quad (\text{B-3})$$

or

$$\bar{\omega}_i^2 [M] \{\phi_i\} = [\bar{K}] \{\phi_i\} \quad (\text{B-4})$$

where the new "stiffness" matrix $[\bar{K}]$ is given by

$$[\bar{K}] = [K] + \omega_0^2 [M] \quad (\text{B-5})$$

Note that Equation (B-4) is the same form as (B-1) and also that the mode shapes $\{\phi_i\}$ have not been effected. Accordingly, the eigenvalue algorithm may now be applied to Equation (B-4), yielding the mode shapes $\{\phi_i\}$ and shifted frequencies $\bar{\omega}_i$; the original frequencies may then be recovered by using Equation (B-2).

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