

# Two-Phase Choked Flow of Cryogenic Fluids in <br> Converging-Diverging Nozzles 

Robert J. Simoneau and Robert C. Hendricks

# Two-Phase Choked Flow of Cryogenic Fluids in Converging-Diverging Nozzles 

Robert J. Simoneau and Robert C. Hendricks<br>Lewis Research Center<br>Cleveland, Obio

## N^S^

National Aeronautics
and Space Administration
Scientific and Technical
Information Branch

## SUMMARY

Data are presented for the two-phase choked flow of three cryogenic fluids nitrogen, methane, and hydrogen - in four converging-diverging nozzles. Oxygen data were reported earlier. The data cover a range of inlet stagnation conditions, all single phase, from well below to well above the thermodynamic critical conditions. In almost all cases the nozzle throat conditions were two phase.

The results indicate that the choked flow rates were not very sensitive to nozzle geometry. However, the axial pressure profiles, especially the throat pressure and the point of vaporization, were very sensitive to both nozzle geometry and operating conditions.

A modified Henry-Fauske model correlated all the choked-flow-rate data to within $\pm 10$ percent. Neither the equilibrium model nor the Henry-Fauske model predicted throat pressures well over the whole range of data. Above the thermodynamic critical temperature the homogeneous equilibrium model was preferred for both flow rate and pressure ratio.

Like the oxygen data, the data of the three fluids could be normalized by the principle of corresponding states.

## INTRODUCTION

The space program involves the storage, handling, and transfer of large quantities of pressurized liquid cryogens. Many of these fluids can be dangerous, and good management of them requires a knowledge of their flow characteristics. Anytime a pressurized liquid cryogen is caused to flow - whether by design or by accident - the potential for vaporization and two-phase choked flow exists. The flow discharge passage can be a variety of geometries: an orifice, a nozzle, a long or short tube, a crack, or a slit. The potential range of fluid conditions is also quite extensive. At the Lewis Research Center an experimental program has been conducted to measure the two-phase choked flow of various cryogenic fluids in a number of geometries over a wide range of initial conditions. The present report documents the work done with convergingdiverging nozzles.

The general field of two-phase choked flow has been well surveyed in references 1 to 3. Reference 4 also contains some good articles on the subject. No attempt will be made herein to review the field. The literature for subcooled inlet conditions is sparse.

Our experiments with cryogens (refs. 5 to 12) include four fluids: nitrogen, oxygen, methane, and hydrogen in nozzles, orifices, tubes, and slits. The experiments cover a wide range of single-phase inlet stagnation conditions. Some of the nitrogen data to be reported are discussed in the literature in references 5 to 7. The only other data on choked flow of subcooled liquids through nozzles are the water data of Sozzi and Sutherland (ref. 13) and Schrock, et al. (ref. 14). There have been a few experiments on subcooled liquids in orifices and tubes. These are cited in references 10 to 12.

Attempts at analyses generally fall into two categories: first, an isentropic homogeneous equilibrium analysis; and second, some attempt to account for thermodynamic nonequilibrium. The isentropic homogeneous equilibrium model is considered standard thermodynamics and is rarely given authorship. An early work would be that of Tangren, et al. (ref. 15). The equilibrium analysis used in this report is summarized in the appendix. A recent paper by Collins (ref. 16) also presents the equilibrium analysis. It is widely accepted that two-phase choked flow experiences some degree of thermodynamic nonequilibrium, mainly in the form of some delay in vaporization as the fluid passes through the saturation pressure. An attempt at describing this phenomenon proposed by Henry and Fauske (ref. 17) is frequently used in nuclear thermal-hydraulic safety analyses. With a small modification the Henry-Fauske model is used herein. This model is also summarized in the appendix. We have had considerable success applying the principle of corresponding states, in addition to these flow models, to twophase choked flows (refs. 18 and 19). This can be a considerable aid in extending the applicability of the data.

In many of these calculations it is important to have a good description of the thermophysical properties. We used a property program called GASP developed by Hendricks, Baron, and Peller (ref. 20).

The present report emphasizes the experiment and the data. The experimental facilities are described in some detail, especially the test sections. All the convergingdiverging nozzle data are tabulated. Fairly extensive data plots are presented to aid in interpretation. Comparisons are made between the data and conventional equilibrium and nonequilibrium analyses. The data presented cover a wide range: four convergingdiverging nozzles; three fluids - nitrogen, methane, and hydrogen; and inlet stagnation temperatures of $0.65<\mathrm{T}_{\mathrm{o}} / \mathrm{T}_{\mathrm{c}}<1.40$, where $\mathrm{T}_{\mathrm{o}}$ is the temperature at stagnation conditions and $T_{c}$ is the temperature at thermodynamic critical conditions. These data are companion data to the oxygen and nitrogen data reported earlier (ref. 8). The data are primarily two-phase choked flow rates and the associated axial pressure profiles for these extensive inlet conditions. Some gaseous nitrogen data are also included both for calibration and as a reference base for data comparisons.

## SYMBOLS

| A | cross-sectional area, $\mathrm{cm}^{2}$ |
| :---: | :---: |
| D | hydraulic diameter, cm |
| G | mass flux, g/cm ${ }^{2}$. sec |
| L | length, cm |
| N | constant, eq. (A19) |
| P | pressure, $\mathrm{N} / \mathrm{cm}^{2}$ |
| r | radius, cm |
| S | entropy, J/g. K |
| T | temperature, K |
| u | velocity, cm/sec |
| v | specific volume, $\mathrm{cm}^{3} / \mathrm{g}$ |
| W | mass flow rate, $\mathrm{g} / \mathrm{sec}$ |
| X | axial distance, cm |
| x | quality |
| Z | compressibility factor |
| $\rho$ | density, $\mathrm{g} / \mathrm{cm}^{3}$ |
| Subscripts: |  |
| b | back conditions |
| c | thermodynamic critical conditions |
| calc | calculated |
| e | thermodynamic equilibrium conditions |
| g | saturated vapor conditions |
| $l$ | saturated liquid conditions |
| max | maximum, choked-flow, conditions |
| meas | measured |
| o | stagnation conditions |
| sat | saturation conditions |
| t | throat conditions |

```
1,2, . . ., n axial stations
Superscript:
*
    flow-normalizing parameter
```


# DESCRIPTION OF EXPERIMENTS 

General Flow Facilities

The experiments were carried out in two separate "once through" cryogenic flow facilities. The facilities are illustrated schematically in figures 1 and 2. They have the same basic characteristics. The essential elements include a low-pressure liquid supply, a high-pressure vessel, a gas-pressurizing system, a primary flowmeter, the test-section assembly, a backpressure valve, a heat exchanger, and a secondary orifice flowmeter near the exit. In addition the high-pressure gas system was arranged so that the gas could be used to warm the liquid. The pressure vessel, primary flowmeter, and test section were enclosed in a vacuum envelope to minimize heat leaks. Fluid pressures and temperatures were measured at appropriate points in the flow system, as indicated. The facility of reference 8 was also of this type. Individual components are discussed in detail later.

The general operation of the flow facility was as follows: After the pressure vessel was filled with liquid from the low-pressure supply, the liquid was warmed to the desired stagnation temperature by bubbling a warm gas of the same type through the liquid. Subsequently, the supply pressure was set by applying the pressurizing gas to the top of the liquid. The stagnation conditions were the pressure and temperature that existed in the inlet mixing chambers, as shown in figures 1 and 2 . Flow was begun by fully opening the backpressure control valve. Once a steady flow was established, choking was demonstrated by varying the backpressure and observing that it did not change the flow rate or throat pressure. From here on the operation of the two test facilities differed, and this difference had a bearing on the results.

In the smaller, 110 -liter, rig shown in figure 1 the procedure was to take discrete data points, normally only one for each fill of the pressure vessel with liquid nitrogen. The object was to maintain all the data points as close as possible to a predetermined stagnation isotherm $T_{o^{*}}$. In reality, since the system would normally drift steadily in temperature, several records of data would be taken for each fill and subsequent flow and then the record closest to the desired isotherm would be selected for data. Also the backpressure was regularly varied throughout the flow so that choking was established for every data point. This procedure produced close control; however, it was very time consuming and used large quantities of fluid.

The larger, 375-liter, system shown in figure 2 was operated differently. Once the steady flow and the choking phenomenon were established, a series of data points were recorded, each at a different stagnation pressure $P_{0}$. The stagnation pressure was normally varied from some high subcooled value down to near saturation for a single tankful. This was done by adjusting the control valve between the high-pressure tank and the test section. This procedure was much faster and more efficient in the use of fluid, but it did result in a wider tolerance on the nominal stagnation "isotherm" than the other procedure. This technique was absolutely essential for the hydrogen and methane data, where fluid cost was high and setup time was long. It also allowed data to be taken for many isotherms near the thermodynamic critical point with nitrogen in order to obtain detailed information on that region.

## Test-Section Assemblies

In these experiments the four converging-diverging nozzle configurations were obtained by using three separate test sections. They are illustrated in figures 3 to 5 . The essential dimensions of these test sections are listed in tables I to III. The fourth nozzle was obtained by flowing the conical nozzle (fig. 3) in both directions.

The test section illustrated in figure 3, as originally designed, had a truncated cone of nominally $7^{\circ}$ half-angle convergence and a cone of nominally $3.5^{\circ}$ half-angle diver gence. It was designated the $7^{\circ}$ conical nozzle. The throat region had a constant-area section 3.2 diameters in length. The transition from the converging cone to the constant-area throat section was smoothed with a radius of curvature of approximately 10 times the throat radius. The transition from the constant-area throat section to the diverging cone was a sharp corner. (With a $3.5^{\circ}$ half-angle cone the term "sharp" can be questioned. It merely means that the cone was machined with a straight wall with no attempt to radius the transition to the constant-area section.) This sharp corner was designated the throat. The test section was instrumented with 15 pressure taps concentrated near the throat. Later in the testing this nozzle was turned around in the rig in order to assess the effect of the convergence angle and approach curvature on the pressure profile. In this orientation the nozzle was designated the $3.5^{\circ}$ conical nozzle.

The test section illustrated in figure 4 was rectangular in cross section. The side walls were parallel. The convergence was effected by linearly tapering one wall at nominally $7^{\circ}$. The divergence was achieved by tapering the same wall at nominally $3^{\circ}$. The opposite wall was straight. The throat region had a constant-area section that was 8.3 times the throat height in length ( $\mathrm{L} / \mathrm{D}=4.15$ ). This constant-area throat section had a width-height ratio of 9.3 . The transitions between sections were the same as for the conical nozzle. The straight wall, which could be considered as the imaginary centerline of a symmetrical nozzle, was instrumented with 12 pressure taps concentrated
near the throat. At three key locations a pressure tap was located on the contoured wall opposite a straight-wall counterpart.

The test section illustrated in figure 5 was a conventional venturi flowmeter and was designed according to the ASME long-radius flow nozzle guidelines (ref. 21). Pressure taps were installed as illustrated. The converging section had a $2: 1$ elliptical curvature that transitioned smoothly into a constant-area section 2.1 diameters in length. The transition from the constant-area throat section to the $4^{\circ}$ half-angle divergence cone was a sharp corner. This sharp corner was designated the throat. Table III gives two values for overall length. The smaller value, 6.80 centimeters, was the distance from the beginning of the elliptical converging section to the end of the $4^{\circ}$ diverging section. The larger number was the distance from the inlet plenum to the beginning of the downstream straight section. The smaller dimension was probably more relevant, since this was really the nozzle shape under consideration. This nozzle was not as heavily instrumented in the throat region as the other two.

The dimensions listed in tables I to III were determined in two ways: They were measured mechanically in an inspection laboratory and were scaled from X -ray photographs of the test-section cross section. The tolerances represented discrepancies between these measurements. One additional measurement of importance was the alinement of the curved and straight walls in the constant-area section of the twodimensional nozzle. From the various measurements we estimated that the misalinement in this area did not exceed 0.002 to 0.003 centimeter of the 0.109 -centimeter height over the 0.905 -centimeter length, a variation of 2 to 3 percent. Because of the very small divergence angle in all the test sections, the location of the exact point of divergence was very difficult to pinpoint, and the axial tap location tolerance was primarily an estimate of this difficulty.

All the nozzles were made of stainless steel with their internal surfaces finished to at least 16 rms . Care was taken to deburr all pressure taps.

In addition to the obvious geometric differences, the four nozzles can be compared in terms of convergence rates. Using the elliptical nozzle and taking the beginning of the ellipse as a reference, the ratio of entrance area to throat area was 14.3 and the nozzle converged to the beginning of the constant-area throat region in 0.751 centimeter. For the $7^{\circ}$ conical nozzle, the convergence from the position where the area ratio was 14.3 to the beginning of the throat region required a distance of 4.14 centimeters. For the $3.5^{\circ}$ conical nozzle, this distance was 7.48 centimeters. For the twodimensional nozzle, this distance was 12.1 centimeters.

The inlet and outlet plenum mixing chambers for these test sections were an important part of the assembly. For the conical and two-dimensional nozzles they were identical and are shown in figure 6. The labyrinth path was designed to avoid any jetting into the nozzle entrance and to break up any stratification in the flow. The cross-sectional area of the innermost passage was $19 \mathrm{~cm}^{2}$, as compared with approxi-
mately $6 \mathrm{~cm}^{2}$ at the inlet of the conical and two-dimensional nozzles. The pressure and temperature measured in the inlet plenum were taken as stagnation conditions. The backpressure and temperature were measured in the outlet plenum. The elliptical nozzle had only an inlet plenum, and it was welded directly to the nozzle as shown in figure 7. The cross-sectional area of the stagnation chamber was $6 \mathrm{~cm}^{2}$, as compared with $0.5 \mathrm{~cm}^{2}$ at the nozzle inlet. There was a mixing chamber in the flow system downstream of the elliptical nozzle (cf. fig. 2) where the backpressure was measured. However, there was a length of straight pipe between the nozzle exit and this chamber.

## Instrumentation

Pressure and temperature sensors. - The only physical measurements made in this experiment were pressure and temperature. Pressures were all measured with straingage transducers. In the 110 -liter rig shown in figure 1 all the static pressures (except the static pressure at the downstream orifice) were measured on a matched set of $689-\mathrm{N} / \mathrm{cm}^{2}$ ( $1000-\mathrm{psig}$ ) transducers rated at $\pm 0.2$ percent of full scale. The differential transducers and the transducer used to measure the downstream orifice static pressure were rated at $\pm 0.3$ percent of full scale. The transducers were calibrated in a standards laboratory and normally exceeded rated accuracy.

The fluid temperatures were measured throughout the flow system, as shown in figure 1, by use of platinum resistance thermometers. The thermometers and accompanying bridge circuits were calibrated in a standards laboratory and were considered accurate to $\pm 0.1$ percent of full scale ( $\pm 0.1$ kelvin). Two thermometers were located in each of the inlet and outlet mixing chambers.

In the 375 -liter rig (fig. 2) a wider range of transducers was used to accommodate the wider range of the experiments undertaken therein. The same accuracy statements made for the 110 -liter rig apply here also. In addition to platinum thermometers in key locations, as shown in figure 2, this rig also had Chromel-constantan thermocouples at various points. They are less accurate than the platinum thermometers and were used as backup measurements.

Flowmeters. - In the smaller rig the flow was measured in two locations, as shown in figure 1, by sharp-edged-orifice flowmeters. The primary flowmeter was located immediately upstream of the test section and metered liquid flow. Because of limited space a special orifice was built, as shown in figure 8. Because of the unusually short entrance region ( $L / D=6$ ) a special fitting that had the same configuration as the actual installation was made for use in calibration. The flowmeter was calibrated in a standards laboratory. According to the resulting calibration curve, flowmetering was considered accurate to within $\pm 0.4$-percent error. Most of the flow rates in the experiment were high enough to be in the extrapolated asymptotic region of the curve. A secondary
or backup orifice flowmeter was located downstream of the test section and the heat exchanger, as shown in figure 1, and metered gas flow.

In the larger rig the flow was metered with a venturi located at the bottom of the dip tube in the high-pressure Dewar, as shown in figure 2. The venturi was approximately 4 meters ahead of the test section. It was a fairly conventional instrument and was calibrated in the same standards laboratory as the flowmeter for the smaller rig. It can be considered accurate to within $\pm 0.3$ percent error. This facility also employed an orifice flowmeter at the exit as a backup or secondary instrument.

Recording and monitoring. - All the data were recorded on a central data acquisition system known as CADDE II, which is described in reference 22. The system uses a solid-state scanner and a four-place integrating digital voltmeter coupled with a binary coded decimal (BCD) encoding unit. It can sample up to 200 inputs at 1 to 40 signals per second, with 20 signals per second being the normal rate. These raw signals were available for immediate playback on a typewriter at the test facility. In addition, the signals were transmitted directly to a time-sharing computer for further processing at the test cell by means of a computer terminal link. Thus, such computed parameters as mass flow rate were rapidly available. The inputs are recorded as percentage of full scale over seven ranges from 10 millivolts to 10 volts. Each signal in succession can be recorded in one of these seven ranges. The overall recording system was rated accurate to within $\pm 0.04$-percent error or $\pm 10$ microvolts, whichever was more accurate. The signals necessary for control and monitoring of the test facility were isolated and amplified through high-impedance differential amplifiers and displayed on panelmounted digital voltmeters.

## Data Accuracy

Accuracy is as much a function of how an instrument is used as it is of its manufactured tolerance. For example, if a $690-\mathrm{N} / \mathrm{cm}^{2}$ transducer is used to measure a $69-\mathrm{N} / \mathrm{cm}^{2}$ pressure, the absolute error remains unchanged but the relative error increases by a factor of 10 . Although this is obvious, it is often overlooked or reported inaccurately. Also some data, such as flow rate, involve the accuracy of both measured data and computed parameters such as fluid density. With these thoughts in mind, two accuracy estimates for each measurement taken in the two facilities (figs. 1 and 2) are listed in table IV. One is the absolute value based on the full range of the instruments used. The other, which is really more useful, is the percentage of error based on the average value over the actual range of the measurement.

The flow -rate measurement depends on the accuracy of computing properties most importantly, density. The thermophysical properties were computed by using a
computer subroutine called GASP, which is described by Hendricks, Baron, and Peller in reference 20. The main component of the program is an equation of state by Bender (ref. 23). Transport properties were refit by the authors of GASP. Bender (ref. 23) gives error estimates of $\pm 0.26$ percent for gas, $\pm 0.18$ percent for liquid, and $\pm 1.47$ percent in the critical region. (The critical region is defined by Bender as $0.66 \rho_{c}$ to $1.5 \rho_{c}$, from the saturation locus to $1.25 \mathrm{~T}_{\mathrm{c}}$.) These error estimates in computing the density of the fluid were included in the error estimates for flow rate presented in table IV.

Because all the pressure readings on the smaller rig were taken on a matched set of $690-\mathrm{N} / \mathrm{cm}^{2}$ transducers, it was possible to condition the data to improve internal consistency. At the beginning of each day's run the system was closed and pressurized to about midrange and a zero-flow pressure record was established. The relation of any individual transducer to the median was remarkably consistent over the many months of the experiment (e.g., transducer 5 was always 0.05 to 0.19 percent above the median). Because of this consistency the computer was programmed to adjust all static pressures by the percentage observed in the daily zero-flow readings. Although this did nothing to improve absolute accuracy, it did help to establish an internal consistency between the axial pressure profile measurements.

Finally, redundant instrumentation was used throughout both facilities. No critical measurements, such as stagnation pressure and temperature or mass flow rate, were made without a backup. Such system checks as flowing with gas and the no-flow check already mentioned were routinely performed.

## RESULTS

Range of Experiments

These experiments investigated the two-phase choked flow of cryogenic fluids in four converging-diverging nozzles. Three separate fluids - nitrogen, methane, and hydrogen - were investigated. The hydrogen and methane data were obtained in the elliptical nozzle. Only nitrogen was investigated in all four nozzles. (The liquid-oxygen and -nitrogen data of ref. 8 were taken in the elliptical and the conical nozzles.) In all cases the initial conditions were single phase. A very extensive range of initial conditions was investigated. The entire investigation is summarized in table V. Throughout the experiments an attempt was made to acquire the data along lines of constant stagnation temperature $T_{0}$; thus, the data are organized along isotherms for presentation.

An interesting way to look at the experimental range is on a temperature-entropy diagram, as shown in figure 9 . Since the process is frequently assumed to be isentropic, this sketch gives a good overview of the location of the stagnation conditions
in relation to the two-phase locus. Figure 9 shows that the range of initial conditions extends from a "hard" liquid to a highly compressible fluid. However, the data are not uniformly distributed throughout the crosshatched region. For example, most of the data have an initial entropy that is less than the critical entropy. Only about 30 of the 545 data runs were at $S_{0} / S_{c}>1.0$. Also only the nitrogen data extended to high temperature. The methane data were below $T_{0} / T_{c}=1.05$ and the hydrogen data were below $\mathrm{T}_{\mathrm{o}} / \mathrm{T}_{\mathrm{c}}=1.0$. Most of the data were choked in the two-phase region; however, some of the high-temperature nitrogen data were sufficiently compressible to choke in the single-phase region above the saturation locus. This is not to say, however, that none of the $S_{o} / S_{c}>1.0$ data were two phase. In fact, 21 of these 30 data runs had throat pressures below the isentropic intercept with the saturation locus. In any case, the overwhelming majority of the data had stagnation conditions that fell into two categories: (1) those of a highly subcooled liquid and (2) those of a very dense but compressible fluid.

## Data Tables

The data from these experiments are presented in tables VI to XIII, with a summary in table V. The data in these tables are described briefly here. The results are discussed in detail later. In all cases in these tables, the "throat" pressure used in the $\mathrm{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}$ column is the pressure read at the tap immediately upstream of the point of divergence.

The data of tables VI to VIII were all taken in the smaller rig (fig. 1) with nitrogen as the fluid. The emphasis was on acquiring data along specific isotherms within close tolerance. The five isotherms - $95,110,119,124$, and 130 K - were chosen to span the region from incompressible liquid to the highly compressible dense fluid near the thermodynamic critical region ( 126.3 K ). The pressures ranged from just above saturation to twice the thermodynamic critical pressure. The two conical nozzles and the two-dimensional nozzle were employed. For the two-dimensional nozzle, pressures 1 to 12 were taken on the straight wall and 13 to 15 were taken on the contoured wall. Measurements 6 and 13,8 and 14 , and 9 and 15 were made directly opposite each other. Emphasis was placed on obtaining a comprehensive set of pressure profiles. For each data set, two readings were taken at two different backpressure levels to demonstrate choking. The reading included in the data tables was that closest to the isotherm. This explains the irregular pattern in the backpressure data. At the end of each of tables VI to VIII an ambient-temperature gas run is included for reference. For the data of table VII the $3.5^{\circ}$ conical nozzle was used (i.e., the $7^{\circ}$ conical nozzle installed backwards). This reversal of flow through the conical nozzle was performed to investigate the influence of the throat turning-radius on the pressure profiles.

Nitrogen data taken in the larger rig (fig. 2) with the elliptical nozzle are presented in table IX. The summary table (table V) shows that the special emphasis was on isotherms near the thermodynamic critical temperature ( 126.3 K ). Also the pressure range was substantial, reaching to almost three times the thermodynamic critical pressure. In most instances the number of data runs per isotherm was not large; however, the isotherms were closely spaced.

Data for methane and hydrogen, also taken using the larger rig with the elliptical nozzle, are presented in tables X and XI, respectively. Because of the handling difficulties and the expense associated with these fuels, the data are somewhat more limited; nevertheless, a substantial range has been covered, especially for methane. The primary purpose of these data is to allow normalized comparisons on corresponding-states principles. The data for the three fluids presented herein, along with the oxygen data already reported (ref. 8), provide a good basis for such comparisons.

The data presented in tables XII and XIII were acquired in a different facility and are summarized in reference 5 . Since they were never documented, they are included herein for convenience and completeness. They have two important features: Some of the points are very close to the thermodynamic critical point, probably closer than any other data in these experiments. Other points are very close to the saturation locus, again probably closer than any other data in these experiments. The experiment, however, was somewhat less precise than the others reported herein. For a description of the experiment, see reference 5 .

Although it was standard practice with the conical and two-dimensional nozzles to establish choking for every run, it was not normal to obtain profiles over a wide range of choked and unchoked backpressures. Two such sets obtained with nitrogen and the 3. $5^{\circ}$ conical nozzle are presented in table XIV. A gas data set is included for reference. The data also offer some insight into system drift. There is a tendency for stagnation temperature to drift upward and, sometimes, for stagnation pressure to drift downward. The 1331~1334 set (table XIV) spanned 3.4 minutes.

## Axial Pressure Profiles

In addition to the data tabulations, the data can be presented in certain graphical forms that are useful for analysis. Primary among them are the axial pressure profiles and plots of the flow rate and the ratio of throat to stagnation pressure as functions of stagnation conditions.

Axial pressure profiles for various conditions are plotted in figures 10 to 16 . The first of these contains nitrogen gas data. Because the nozzles were of varying lengths and all had constant-area sections separating convergence from divergence, a special
abscissa had to be invented to normalize the plots. The pressures in the converging and diverging sections are plotted as a function of area ratio $A / A_{t}$. The pressures in the constant-area section are plotted as a function of the fraction of that section where the pressure was measured $\mathrm{X} / \mathrm{L}$. (Note that the point of divergence, the nominal throat, is located at $X / L=1$.) All the two-phase pressure profiles have one common characteristic, which is a function of the tendency of the system to drift upward in temperature during operation. From figure 9 it is clear that such a drift in stagnation temperature at constant stagnation pressure will increase the isentropic intersection with the saturation locus. Also, the increased average temperature will decrease average density. It is known that the saturation (vaporization) pressure is closely related to the throat pressure in two-phase choked flow. Thus, the combination of decreased pressure difference between stagnation and saturation, as well as decreased density, will cause a drift downward in flow rate as the stagnation temperature increases.

## Flow Data Plots

The choked flow rates and the ratios of throat to stagnation pressure are plotted for each of the four nozzles in figures 17 to 24 . Figures 17 and 18 present all the data from table VI for the $7^{\circ}$ conical nozzle. Most of the data fall on the listed isotherm to within $\pm 0.3$ kelvin. The symbols for the data that fall outside this tolerance are tailed in the figures. If the tail is on the bottom of the symbol, the temperature is low, and conversely. Figures 19 and 20 present the data from table VIII for the two-dimensional nozzle. All the remarks made for figures 19 and 20 apply to these data as well. The isotherms for the two nozzles are quite close in value and thus should be good for comparisons. Figures 21 and 22 present the data from table VII for the $3.5^{\circ}$ conical nozzle. Only two isotherms were run for this nozzle. Again, the same remarks concerning tolerances and plotting apply. In general, the conical-nozzle data have the closest temperature tolerance.

The data in figures 23 and 24 are for the elliptical nozzle and are drawn from two sources. They include the data of table IX, open symbols, and also the data of table XIII, solid symbols. There is some mismatch in the isotherms and the tolerances are greater; thus the stagnation temperatures shown on the figures should be treated as nominal and the tables should be consulted for specific detail. The one set was generally at low pressure, and the other was at high pressure. Both sets are needed to get a full picture of the performance of the nozzle.

## DISCUSSION OF RESULTS

## Choked Flow Rates and Pressure Ratios

The data for nitrogen taken in the four separate nozzle configurations and presented in figures 17 to 24 are summarized in figures 25 and 26 . The isotherms selected for summary were ones where there was reasonable correspondence from nozzle to nozzle. (Only the 110 and 119 K isotherms include data from all four nozzles.) The lines faired through the data in figures 17 to 24 are reproduced in figures 25 and 26 . The values of the isotherms are much more nominal in these figures, and they are used primarily for qualitative remarks. See figures 17 to 24 and the data tables for quantitative details.

Beginning with the flow-rate data (fig. 25), the first observation is that there is strong agreement between the data from the various nozzles in most regions of the experimental parameters. In general the flow-rate data from the two-dimensional nozzle fall a few percent below those from the $7^{\circ}$ conical nozzle, which in turn fall below those from the elliptical nozzle, over the whole range of the experiment. An exception occurs along the 124 and 130 K isotherms at the low-pressure end where the curves fold over. This small change in trend in these regions may reflect a greater sensitivity to geometry near the thermodynamic critical point, where density gradients are very steep. The biggest difference from nozzle to nozzle occurs at the low-pressure end of the 110 and 119 K isotherms. Here, in the most significant case, the difference in flow rate was as much as 34 percent between nozzles. This is also the region of greatest differences in the ratios of throat to stagnation pressure. This is discussed in conjunction with the axial pressure profiles in the next section.

The summary plots of the ratio of throat to stagnation pressure as a function of stagnation conditions (fig. 26) exhibit much less agreement from nozzle to nozzle. The question is whether this is a nozzle geometry effect, or whether it merely reflects the difference in the location of the nozzle "throat" pressure tap. The data appear to suggest both. For example, from the gas pressure profiles (fig. 10), the two-dimensional nozzle would be expected to read a higher "throat" pressure than the conical nozzle, but only by about 8 percent. At $T_{o}=130 \mathrm{~K}$ and $\mathrm{P}_{\mathrm{o}}=600 \mathrm{~N} / \mathrm{cm}^{2}$ this is true, with the difference being about 6 percent; however, at $T_{o}=95 \mathrm{~K}$ and $\mathrm{P}_{\mathrm{O}}=600 \mathrm{~N} / \mathrm{cm}^{2}$, the twodimensional nozzle's throat pressure was almost 100 percent above that of the conical nozzle. It would appear that in this region the pressures are very sensitive to geometry.

Axial Pressure Profiles
Gas profiles. - Pressure profiles from the gas runs were normalized and are plotted in figure 10. In most of the cases shown, profiles for two separate backpres-
sures are plotted. It is clear from the data of figure 10 that all four nozzles performed similarly in gas flows. The converging-region profiles are almost identical, even in the very sensitive throat region. They could all be readily choked. The measured flows all range from 96 to 98 percent of the computed isentropic expansion.

On the other hand, there are small differences and deviations that should be noted to aid interpretation of the data. For example, although there was good overall profile agreement, there were differences in the "throat" pressures from nozzle to nozzle. The throat pressure measured in the $7^{\circ}$ conical nozzle was in perfect agreement with the calculated throat pressure. However, this pressure tap was 0.185 centimeter ahead of the point of divergence, which was designated as the throat. Thus, it could be reading a little low, or the throat location could be slightly in error. Actually, extrapolations of the "throat" pressure readings of the $3.5^{\circ}$ conical nozzle and the twodimensional nozzle to the point of divergence are closer to the computed value. Although these differences are small, they should be considered when evaluating the choked-flow pressure ratio data.

The gas profiles serve as a convenient reference for discussing the two-phase-choked-flow data. If the flow were ideal, there would be no pressure drop in the constant-area flow region. However, despite only a 3 to 4 percent deviation from ideal flow, all the nozzles show strong linear pressure drops in this region, roughly 10 percent of the total. The pressure profiles in the constant-area region for the $7^{\circ}$, roundedentrance and the $3.5^{\circ}$, sharp-corner-entrance conical nozzle geometries are nearly parallel. This suggests that the strong pressure drop was not the result of some small geometric differences from nozzle to nozzle but was, in fact, phenomenological. Computing the friction in the entrance of a tube for the conditions of figure 10 indicates a friction pressure drop of about $8 \mathrm{~N} / \mathrm{cm}^{2}$, or about 2 percent of the total nozzle pressure drop. If it were assumed that the 3 - to 4 -percent flow deviation from ideal was due to the area change from this same boundary-layer growth, the required 3-to 4 -percent area change would produce a 6 - to 8 -percent pressure drop. Thus, a friction boundary layer in the throat that was only 1.5 to 2.0 percent of the throat radius could produce pressure drops of the order shown in figure 10.

In summary, the gas profiles show that a small constant-area section of 2 - to $4-\mathrm{L} / \mathrm{D}$ length in the throat region of a converging-diverging nozzle produces a strong pressure gradient. However, the two-phase situation is much more complex.

Two-phase profiles. - With the gas behavior as background, the pressure profiles associated with the two-phase choked flow of subcooled nitrogen can be examined. The first subject of interest from the two-phase profiles is the nature of the choking phenomenon. This can be described by reference to a series of axial pressure profiles in which the flow was both choked and unchoked. Such a series was obtained for the $3.5^{\circ}$ conical nozzle (fig. 11). The distinguishing feature of the data in figure 11(a) is that the stagnation pressure is substantially above the saturation pressure. The average values
of both the isothermal and isentropic saturation pressures are indicated in figure 11(a). A band is shown because these values drift with stagnation temperature. See the appendix for further discussion of the saturation pressures.

The profiles in figure 11(a) are all remarkably similar up to the diverging end of the constant-area throat section. Since $P_{0}$ was the same for all readings, the mass flux data $G$ (see fig. 11(a) key) indicate that the first two profiles (readings 1331 and 1332) are unchoked. Also, they are probably for an all-liquid phase since the lowest pressure is $40 \mathrm{~N} / \mathrm{cm}^{2}$ above saturation. The profiles for readings 1333 and $1334 \mathrm{ap}-$ pear to be choked. (The small difference in $G$ is the drift previously discussed.) The pressure drops in the constant-area region were very similar in all four cases. This led to the conclusion that the flow was dominated by liquid to the exit end of the constant-area region.

The profiles in figure 11(b) present a different picture. They are characterized by the stagnation pressure being close to saturation. The profile for the first reading (1350) is very similar to the profiles in figure 11(a). Depending on which thermodynamic path was selected, the constant-area throat pressures were either slightly above or slightly below the saturation pressure. The flow was not choked. The remaining four profiles are clearly different, and in every case the pressures in the constant-area region were clearly below saturation regardless of the thermodynamic path. A profile for nitrogen gas (reading 1344) is also shown in figure 11(b). In the constant-area region the gas pressure distribution was very similar to that for the four two-phase profiles ( 1351 to 1354 ). This profile similarity certainly seems to imply that vapor dominated the constant-area region for all these nozzle profiles except the first reading (1350).

The combination of the data shown in figures $11(\mathrm{a})$ and (b) strongly implies (1) that for $P_{o}$ near $P_{S a t}\left(S_{o}\right)$ the profiles show a fairly steep pressure drop in the constantarea region and reflect a vapor-dominated flow; and (2) that for $P_{o}$ well above $P_{\text {sat }}\left(S_{o}\right)$ the profiles have a very strong initial gradient $d P / d x$ and then almost level off in the constant-area region and appear to reflect a liquid-dominated flow to the point of divergence.

The three profiles in figure 12 give us a little closer look at the effect of subcooling. They are axial pressure profiles for a range of stagnation pressures from slightly above saturation to three times the saturation pressure. In the first case (triangles), the flow expanded below the saturation pressure well upstream of the throat; in the second case (squares), the saturation pressure appeared to occur right at the entrance to the throat region. In both cases the pressure drop in the constant-area (throat) region is similar to the gas profiles. This would suggest that vaporization occurred at, or upstream of, the throat and that vapor was present and dominant in the constant-area region. In the final case (circles), the isentropic saturation pressure coincided with the pressure at the last two taps in the throat. The shape of the profile and the fact that all
pressures were above saturation strongly suggest that the flow was liquid dominated to the exit of the throat region and that any vapor was present in very small quantities.

Another way of looking at the pressure is in terms of what the pressure would ideally be if there were no vapor present (i.e., all-liquid flow). For the measured G, a pressure was computed that would produce that mass flux if the flow were all liquid. The points are shown in figure 12. In each case this computed pressure was very close to the pressure measured at the entrance to the constant-area region and substantially above the value nominally designated as the throat pressure. In the two cases where vapor was clearly present (readings 1546 and 1531), this is not too surprising; but in the case where liquid appeared to dominate the flow (reading 1584) something, either a vena contracta or a small amount of vapor, must be constricting the flow.

Another point of interest is the shape of the profiles in the diffuser, particularly in the first two cases (readings 1546 and 1531). Further downstream the profiles are virtually identical; however, just downstream of the throat there are substantial differences. This could represent separation or it could be the result of some nonequilibrium behavior.

As has been pointed out, so that we could explore questions of nozzle geometry, the $7^{\circ}$ conical nozzle was turned around and installed in reverse to its normal flow direction, forming the $3.5^{\circ}$ conical nozzle (cf. fig. 3). The difference between the two nozzle configurations was small. One nozzle had a $7^{\circ}$ convergence with a rounded throat entrance and a sharp divergence at $3.5^{\circ}$. The other nozzle was the reverse of this. At these small angles, the difference between "sharp" and "rounded" is more a matter of specification on a drawing than it is a fact. Thus, we are talking about the effect of small differences. Selected isotherms were repeated. Profiles from two of these cases are shown in figures 13 and 14. In both cases the data are characterized by very close control on the stagnation parameters.

In the first case (fig. 13) the profiles and flow rates for the two flow directions are very similar. The small differences may be significant in terms of mechanism, but they make little difference in the results. On the other hand, at stagnation conditions closer to saturation the two nozzle configurations produced a substantial shift in the pressure profiles and a 10 -percent difference in flow rate. Thus, it would appear that, if the potential for vaporization is present (such as in fig. 14 with a relatively high saturation pressure), the small differences in nozzle configuration can be significant. In the more highly subcooled case (fig. 13), the differences are less significant, at least in their effect on the flow rate.

The same phenomena appear when all the nozzle profiles are compared in a single plot, such as shown in figures $15(\mathrm{a})$ and (b). In figure 15(a), for example, with highly subcooled conditions the pressure profile in the constant-area (throat) region varies in shape from nozzle to nozzle; however, the resulting "throat" pressure and flow rate are not significantly affected. On the other hand, when the saturation pressure is
higher, as in figure $15(\mathrm{~b})$, the pressure profiles are even more varied and the influence on "throat" pressure and flow rate is strong. These variations in sensitivity have already been pointed out in the overall summary in figures 25(a) and 26(a).

Before we leave this subject, one final point - drawn from figures 11 to 15 - is in order. The term "throat" has become very imprecise for these flows. Recall from the test-section descriptions that the constant-area regions were introduced to ensure a pressure tap at the throat. It was initially expected that very little vapor would exist upstream of the point of nozzle divergence; consequently, negligible pressure drop was expected in the constant-area region. The data show this to be true only in the highly subcooled case; but, as $P_{o}$ approached $P_{\text {sat }}\left(S_{0}\right)$, the small 2- to 4-L/D constantarea section played a role in the pressure drop. The point of vaporization, the amount of vapor, and the location of the "throat" all seem to depend on relative subcooling. This makes consistent reporting difficult and offers a complex challenge to analytical modeling.

The effect of varying stagnation temperature on the pressure profiles in the twodimensional nozzle is shown in figure 16. For this plot the pressure difference $P_{o}-P_{\text {sat }}\left(S_{0}\right)$ was held roughly constant; thus, the flow was initiated at roughly the same distance from the saturation locus for each case. The first and most obvious observation is that the shift in the profiles follows the trend in the saturation pressures. This would suggest that perhaps the flow could be predicted if the isentropic saturation pressure were known. Although in a gross sense this is probably true, a close examination reveals that it is not quite so simple. If the pressure data in the constant-area throat are linearly extrapolated to the point of divergence ( $X / L=1$ ), the trend is not consistent with respect to $P_{\text {sat }}\left(S_{0}\right)$. For the three highest temperature isotherms this extrapolated pressure is below $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{S}_{\mathrm{o}}\right)$. For the 109.8 K isotherm it almost coincides with $P_{S a t}\left(S_{o}\right)$. And for the lowest temperature isotherm, 95.8 K , the extrapolated pressure is above $P_{\text {sat }}\left(S_{0}\right)$. Thus, in terms of modeling, the isentropic pressure is not a simple barometer of the flow characteristics.

Another interesting observation from figure 16 is that the profile for the 129.9 K isotherm, which is greater than $\mathrm{T}_{\mathrm{c}}=126.3 \mathrm{~K}$, has the same shape in the converging region and the throat area as the other four isotherms. This suggests that the expansion was isentropic or nearly isentropic (definitely not isothermal). The initial entropy for this case was $S_{o} / S_{c}=0.833$; and thus, like the other isotherms in figure 16 , the expansion was through the liquid locus into the two-phase region.

The pressure profiles from the two-dimensional nozzle also offer some insight into the validity of a one-dimensional flow model. Figure 4 and table II show that the twodimensional nozzle was really a half nozzle with one wall straight and one wall contoured. Most of the pressure taps were on the straight wall but at three locations: one just upstream of, one in, and one just downstream of the constant-area throat region. Pressure taps were placed in the contoured wall, nominally opposite its straight-wall
counterpart (cf. table II). At the upstream location the contoured-wall pressure was consistently 2 to $6 \mathrm{~N} / \mathrm{cm}^{2}$ higher than the straight-wall pressure; however, it was measured slightly farther upstream. In the constant-area region the nominal "throat" tap frequently read the same as its contoured-wall counterpart. Occasionally, the contoured-wall pressure was 1 to $3 \mathrm{~N} / \mathrm{cm}^{2}$ higher. In this case, however, the contoured-wall tap was slightly farther downstream. Just downstream, the pressures were virtually identical again, with the contoured-wall pressure occasionally slightly higher, at 1 to $2 \mathrm{~N} / \mathrm{cm}^{2}$. In this case, the contoured-wall tap was slightly upstream, and again position alone could account for these small differences. Also remember that the error band on the pressure transducers was $\pm 1.4 \mathrm{~N} / \mathrm{cm}^{2}$. The evidence in these pressure profiles testifies strongly to the validity of one-dimensional modeling.

On the other hand, in the two-dimensional nozzle there were no precipitous pressure drops coming into the constant-area region, such as observed in the conical nozzle at high subcooling. Thus, it is possible that separation at the inlet (i.e., twodimensional effects) could be occurring in one nozzle and not the other. This would indicate an extreme sensitivity to small machining differences since both nozzles converged and diverged at the same angles.

## Throat Pressure Anomaly

Returning to the summaries of the ratios of throat to stagnation pressure (fig. 26), we observe an anomalous flat region, particularly along the 110 and 119 K isotherms. As is shown in the next section, the commonly used theories do not describe this flat region. This presents problem to the analyst. Although the effect was most prominent in the conical nozzle, it appears in the data of all the nozzles.

Presenting the "throat" pressure data in the manner of figure 26, although it is based on a long-standing tradition, tends to mask the trend since $P_{o}$ appears in both the ordinate and the abscissa. Figure 27 is a plot of "throat" pressure as a function of stagnation pressure for the conical nozzle in the reversed-flow orientation along the 119.3 K isotherm. This nozzle and isotherm were chosen because the data exhibit the trend well and the isotherm was in extraordinarily close tolerance, with all points being within 0.2 kelvin of 119.3 K . The variation of the isentropic saturation pressure with stagnation pressure was plotted for reference. The isothermal saturation pressure was also marked. Beginning at the low-pressure end (the first stagnation data point is only $6 \mathrm{~N} / \mathrm{cm}^{2}$ above saturation), the "throat" pressure decreased to a minimum at around $P_{o}=305 \mathrm{~N} / \mathrm{cm}^{2}$ and then increased steadily until at high stagnation pressure it began to merge along the isentropic saturation pressure locus.

In figure 28, data for the $7^{\circ}$ conical nozzle are added to those for the $3.5^{\circ}$ conical nozzle. The isothermal tolerance had to be increased to $\pm 0.3$ kelvin in order to get
enough data points. The general trend for the $3.5^{\circ}$ nozzle was strongly reinforced, although the minimum point shifted slightly to $P_{o}=330 \mathrm{~N} / \mathrm{cm}^{2}$. This could be related to a small difference in the physical location of the "throat" pressure tap ( -0.185 cm in the $7^{\circ}$ nozzle and -0.083 cm in the $3.5^{\circ}$ nozzle).

A careful study of the axial pressure profiles (tables VI(c) and VII(b)) in the manner of figure 12 revealed that the pressure profiles underwent changes in shape, especially in the constant-area region. These shape changes corresponded to the slope changes in "throat" pressure in figure 28.

Data points from the two-dimensional and elliptical nozzles that fall within the isothermal tolerance of $\pm 0.3$ kelvin are added in figure 29. Although these points tend to support the overall trend, the variance from the conical-nozzle data would weaken any phenomenological conclusions we might draw, especially with regard to the location of the minimum and the correspondence to the isentropic saturation pressure. Nevertheless, the general trend does appear to be phenomenological and presents real problems for any predictive models.

## Comparison of Theory with Data

The theoretical models for two-phase choked flow in most common use today are the homogeneous equilibrium model and a nonequilibrium model developed by Henry and Fauske (ref. 17). The isentropic, homogeneous equilibrium expansion is really rather basic fluid mechanics and thermodynamics and is rarely given authorship today. The equations as they apply to the two-phase choked flow of cryogenic fluids were presented some time ago by the authors in reference 6. They are summarized in the appendix. The model of Henry and Fauske (ref. 17) is particularly popular because, since it was developed as a departure from equilibrium, equilibrium calculations can be used as a base. For comparison with the data herein the Henry-Fauske calculation was modified to have an isentropic path in the single-phase part of the expansion from stagnation to saturation. Thus, the saturation pressure corresponds to the stagnation entropy $S_{0}$. This is also discussed in the appendix.

The flow-rate summaries of figure 25 are compared with the two theories in figure 30. The mean lines of the data from the four nozzle configurations fall in the crosshatched bands shown in figure 30. Part of the spread is due to variations in geometry and part is due to slight differences in stagnation temperature from nozzle to nozzle. The calculations were carried out along stagnation isotherms, as indicated in figure 30. Detailed comparisons are not warranted herein but can be made by going to figures 17, 19, 21, and 23.

The first, and most important, observation from the data is that the modified Henry-Fauske model correlates all the flow-rate data herein to within $\pm 10$ percent.

However, keeping in mind that in general the flow-rate data are well correlated, there are two very obvious deviations. First, the theory appears to consistently overpredict at high stagnation pressures (i.e., high subcooling). Second, the data and theory slopes are slightly different. The flow discrepancy can be at least partially accounted for in the nonideal nature of the nozzles. Recall that the ratios of the measured to ideal gas flow for the conical and two-dimensional nozzles were 0.96 to 0.97 (cf. fig. 10). At $P_{o}=650 \mathrm{~N} / \mathrm{cm}^{2}$, for example, the theory overpredicts the average of the data by a consistent 7 percent. Thus, if the nonideal correction were applied to the theory, the correlation would be quite accurate. This type of calibration, of course, does not identify the source of the departure from ideal one-dimensional flow; it merely quantifies it. For the nozzles in question the most likely candidate is a small boundary-layer buildup in the constant-area region. Another possibility would be a poor radius entering the constant-area region. It has also been pointed out that it is not clear where vaporization begins. Thus, taking the flow into account, both theories predict the high-stagnation-pressure (i.e., high subcooling) data quite well.

The slope difference is a little more complex. Let us examine first the 95 K isotherm. The modified Henry-Fauske nonequilibrium model consistently overpredicts the data by about 7 percent over the whole isotherm; thus, it predicts very well over the whole isotherm when the flow coefficient is considered. The equilibrium model does not do this. It substantially underpredicts as $P_{o}$ approaches $P_{\text {sat }}\left(S_{0}\right)$. This is discussed more later. When the throat-pressure anomaly appears, such as along the 119.5 K isotherm, the slope change is no longer explainable in terms of flow coefficient, and both models begins to underpredict. As discussed earlier this seems to be a physical phenomenon in the flow, and it is clear from the data trends that neither model comprehends it.

We see from figure 30 that the two models run parallel and only a couple percent apart at high $\mathrm{P}_{\mathrm{o}}$. Then, as the stagnation pressure decreases, they begin to diverge until, as $P_{o}$ approaches $P_{\text {sat }}\left(S_{o}\right)$, the equilibrium prediction may be 50 percent below the nonequilibrium. This deviation tends to be greater along the lower temperature isotherms, where the single-phase fluid is nearly incompressible. As $T_{o}$ increases toward the thermodynamic critical temperature the two models tend to come closer until, along the 130 K isotherm, they have almost completely converged. In fact, at this point the equilibrium model is more accurate, as shown in the following discussion on pressure. The nonequilibrium model of Henry and Fauske really only makes sense for $\mathrm{S}_{\mathrm{o}}<\mathrm{S}_{\mathrm{c}}$, and thus is restricted accordingly.

The ability of the theoretical models to predict ratios of throat to stagnation pressure (fig. 31) is much less impressive. First, of course, the data themselves show a much wider variation from nozzle to nozzle. Part of this is the result of the "throat" pressure-tap location being slightly different for each nozzle, as discussed earlier. More likely though, it reflects a sensitivity to geometry. The one-dimensional anal-
yses, however, make no recognition of geometry and thus cannot account for these variations. This is especially true in the anomalous flat regions most prominent along the 110.0 and 119.5 K isotherms. Although the analytical models cannot be expected to reflect geometry variations, they should show proper trends. Both models substantially overpredict pressure ratios near saturation. At low stagnation pressures the fluid passes through the saturation pressure substantially farther upstream than the models predict.

An exception to all this is the prediction by the homogeneous equilibrium model along the 130.0 K isotherm. The prediction of flow rate and pressure ratio is excellent. The peak in the pressure ratio corresponds to the point where the throat pressure begins to be less than the saturation pressure. Where $P_{t} / P_{o}$ peaks, the computed throat pressure is the saturation pressure. In the stagnation pressure region above the point where $P_{t} / P_{o}$ peaks, the volumetric expansion due to vaporization is so great that it chokes immediately. Colins (ref. 16) refers to this as "discontinuous choking." In the stagnation pressure region below the point where $P_{t} / P_{o}$ peaks, the pressure at the throat is predicted to be below saturation. The data agree well with this prediction. If the calculation is continued, the sharp change occurs as $S_{o}$ passes through $S_{c}$ and the expansion begins to pass through the vapor locus rather than the liquid locus. Finally, the calculation converges to a single-phase choking and the expected $\mathrm{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}=0.53$ for gaseous nitrogen (i.e., a diatomic ideal gas).

In general, there are few data to verify these trends; however, a couple of data points from the two-dimensional nozzle tend to support them. The first four runs in table VII(e) have entropies in excess of the critical entropy, and the pressure ratio is dropping in the manner predicted by the homogeneous theory. The other isotherms computed by the equilibrium model also show the peak, but none of the data along those isotherms support it.

In general, on the liquid side ( $S_{o}<S_{c}$ ) the nonequilibrium model of Henry and Fauske (ref. 17), as modified herein, is more reliable than the equilibrium model for stagnation temperatures below the thermodynamic critical temperature, especially near the saturation locus. For temperatures at or above $T_{c}$ the homogeneous equilibrium model is preferred. For $S_{o}>S_{c}$, only the homogeneous equilibrium model is applicable.

## Data Normalization by Corresponding-States Principle

In addition to the extensive investigation of nitrogen, methane and hydrogen were also explored by using the elliptical nozzle in the larger rig (fig. 2). One reason for doing this was to see if the principle of corresponding states could be applied to choking flow. Basically, the principle of corresponding states says that a given thermophysical property of various fluids can be generalized onto a single curve through normalization
by the appropriate critical constants of the fluid. Although this is a well-established principle for static equilibrium properties, it has not been used for the correlation of dynamic flow parameters. The choice of fluids was based on an interest in handling hy drogen and methane (or liquified natural gas) as liquids and on a unique capability for handling such fuels that existed in the larger rig.

It was fairly straightforward to derive a flow normalization parameter (ref. 18)

$$
\mathrm{G}^{*}=\sqrt{\frac{\rho_{\mathrm{c}} \mathrm{P}_{\mathrm{c}}}{\mathrm{Z}_{\mathrm{c}}}}
$$

However, it was quite difficult to obtain data over a wide range in all three fluids along lines of constant reduced temperature $T_{0} / T_{c}$. A feel for the problem can be obtained by examining the range of critical parameters in table XV. It simply was not possible to obtain fine control over this whole range in a single facility.

The data that best met constant reduced isotherms were selected from tables IX to XI and are plotted for reduced choked flow rate in figure 32 and for ratios of throat to stagnation pressure in figure 33. Three symbols were used to identify $T_{o} / T_{c}$ levels, and three shadings to represent the various fluids. Although the data are limited, the results in figures 32 and 33 appear to justify the application of the corresponding-states principle to two-phase choked flow. Data over a very wide area are brought tightly together on a single plot. To appreciate the extent to which the flow-rate curves have been collapsed, we must examine the critical constants in table XV. For the three fluids, $G^{*}$ varies by a factor of 5 . Most of the deviations can be explained in terms of stagnation temperature deviations from the nominal isotherm. The G* normalization appears to be correct.

In separate papers (refs. 18 and 19) all the data of tables IX to XI were smoothed, crossplotted, and compared with the homogeneous equilibrium analysis for a more complete study. This study supported these results more completely. Subsequent work in another facility with oxygen and nitrogen (ref. 8) mapped a wide range with very close control and established the principle convincingly. In the case of hydrogen, because of its quantum nature, the normalization parameters for corresponding states are different, and $G^{*}$ must be modified to include a function that covers the full temperature range (ref. 19). The essential message herein is that corresponding-states normalization works for two-phase choked flow.

## SUMMARY OF RESULTS

Experiments were conducted to investigate two-phase choked flow of subcooled cryogens in four converging-diverging nozzles using three separate fluids. There were
three axisymmetric nozzles: one with a $7^{0}$ half-angle conical convergence, one with a $3.5^{\circ}$ half-angle conical convergence, and one with a $2: 1$ elliptical convergence. The fourth nozzle was two dimensional with a $7^{\circ}$ convergence. The primary fluid investigated was nitrogen. The other two fluids were methane and hydrogen.

Data were acquired over a range of stagnation conditions:
(1) $0.65<\mathrm{T}_{\mathrm{o}} / \mathrm{T}_{\mathrm{c}}<1.40$
(2) $0.20<\mathrm{P}_{\mathrm{o}} / \mathrm{P}_{\mathrm{c}}<2.80$ (4.50 for hydrogen)
(3) $0.30<\mathrm{S}_{\mathrm{o}} / \mathrm{S}_{\mathrm{c}}<1.45$
where $T_{0}, P_{0}$, and $S_{0}$ are the temperature, pressure, and entropy at stagnation conditions and $T_{c}, P_{c}$, and $S_{c}$ are these measurements at thermodynamic critical conditions. The data were not uniformly distributed over these ranges. In general, the stagnation conditions can be classified into one of two categories: (1) a highly subcooled liquid or (2) a very dense but compressible fluid. In almost every case the throat conditions were two phase. For the most part, data were acquired parametrically along lines of constant stagnation temperature.

The report includes tabulations and selected plots of data from 545 separate runs over these ranges. The primary data acquired were the choked-flow rates and the accompanying nozzle pressure profiles.

The major results of the data are as follows:

1. Along a given stagnation isotherm the choked flow rates were not strongly variant from nozzle to nozzle. On the other hand, the ratio of throat to stagnation pressure was very sensitive to nozzle geometry.
2. The axial pressure profiles of the various nozzles indicate that the pressure at which vaporization occurred was very sensitive to geometry and initial conditions.
3. The variation in throat pressure as a function of stagnation pressure along a given isotherm was not monotonic. At low, near saturation, stagnation pressures the throat pressure decreased as stagnation pressure increased. At intermediate stagnation pressures the throat pressure increased with stagnation pressure, yielding an almost constant ratio of throat to stagnation pressure. Finally, at high stagnation pressures the throat pressure was relatively constant near the isentropic saturation value.
4. A modified Henry-Fauske nonequilibrium model correlated all the choked-flowrate data to within $\pm 10$ percent and is recommended for $T_{o}<T_{c}$. Homogeneous equilibrium calculations were more accurate above the thermodynamic critical temperature and are recommended for that region. No model did a particularly good job of predicting throat pressure over the whole range of the experiment; however, the equilibrium model was very good above the thermodynamic critical temperature.
5. The flow-rate and pressure-ratio data for the three fluids investigated nitrogen, methane, and hydrogen - could be normalized to universal curves by the thermodynamic critical constants in accordance with the principle of corresponding states.

## Lewis Research Center,

National Aeronautics and Space Administration, Cleveland, Ohio, January 4, 1979, 506-21.

The basic equations for two-phase choked flow in a nozzle are quite straightforward. They are summarized here for the convenience of the reader. The equations presented are for homogeneous one-dimensional flow. Simply, this means that the flow is uniform across the nozzle cross-section and that both phases are traveling at the same velocity. The equations are more frequently derived without the latter assumption, allowing each phase to have a different velocity; however, this makes the presentation more complex and, in practice, the velocities are assumed to be equal anyhow. Since this was the case in the present work, the two velocities are assumed to be equal at the outset and are denoted by $u$. Friction is neglected because of the high acceleration that occurs in a nozzle.

Under these assumptions the one-dimensional momentum equation in the absence of friction is

$$
\begin{equation*}
-\mathrm{AdP}=\mathrm{d}\left[\mathrm{u}\left(\mathrm{~W}_{\mathrm{g}}+\mathrm{W}_{l}\right)\right] \tag{A1}
\end{equation*}
$$

With the definition for mass flux

$$
\begin{equation*}
G=\frac{W}{A} \tag{A2}
\end{equation*}
$$

and with the one-dimensional continuity equation

$$
\begin{equation*}
\mathrm{W}=\mathrm{W}_{\mathrm{g}}+\mathrm{W}_{l}=\text { Constant } \tag{A3}
\end{equation*}
$$

equation (A1) becomes

$$
\begin{equation*}
-\frac{1}{G}=\frac{d u}{d P} \tag{A4}
\end{equation*}
$$

To express $u$ in terms of measurable quantities, we introduce the concept of quality (fluid vapor fraction).

$$
\begin{equation*}
\mathrm{x}=\frac{\mathrm{w}_{\mathrm{g}}}{\mathrm{~W}} \tag{A5}
\end{equation*}
$$

where

$$
\begin{equation*}
\mathrm{W}_{\mathrm{g}}=\frac{\mathrm{uA} \mathrm{~A}_{\mathrm{g}}}{\mathrm{v}_{\mathrm{g}}} \tag{A6}
\end{equation*}
$$

and manipulate

$$
\begin{aligned}
x G v_{g} & =u\left(\frac{A_{g}}{A}\right) \\
& =u\left(1-\frac{A_{l}}{A}\right) \\
& =u\left(1-\frac{W_{l} v_{l}}{u A}\right) \\
& =u\left\{1-\left[\frac{(1-x) W v_{l}}{u A}\right]\right\} \\
& =u-(1-x) v_{l} G
\end{aligned}
$$

Thus

$$
\begin{equation*}
\mathrm{u}=\mathrm{G}\left[\mathrm{xv}_{\mathrm{g}}+(1-\mathrm{x}) \mathrm{v}_{l}\right] \tag{A7}
\end{equation*}
$$

The term in brackets on the right side of the equation is recognized as the definition of the two-phase specific volume

$$
\begin{equation*}
\mathrm{v}=\mathrm{xv}_{\mathrm{g}}+(1-\mathrm{x}) \mathrm{v}_{l} \tag{A8}
\end{equation*}
$$

After this definition of the two-phase specific volume is substituted into equation (A7), the momentum equation (A4) becomes

$$
\begin{equation*}
-1=G \frac{d(v G)}{d P} \tag{A9}
\end{equation*}
$$

Multiplying by $\mathbf{v}$ gives

$$
-v=v G \frac{d(v G)}{d P}
$$

or

$$
\begin{equation*}
-\mathrm{v}=\frac{1}{2} \frac{\mathrm{~d}(\mathrm{vG})^{2}}{\mathrm{dP}} \tag{A10}
\end{equation*}
$$

Equation (A10) can then be integrated, subject to the condition that $G=0$ when $P=P_{0}$, as

$$
\begin{equation*}
G^{2}=-\frac{2}{v^{2}} \int_{P_{o}}^{P} v d P \tag{A11}
\end{equation*}
$$

Carrying out the differentiation in equation (A9) yields

$$
\begin{equation*}
-1=G\left[v \frac{d G}{d P}+G \frac{d v}{d P}\right] \tag{A12}
\end{equation*}
$$

The choked-flow criterion is

$$
\begin{equation*}
\left.\frac{\mathrm{dG}}{\mathrm{dP}}\right]_{\mathrm{t}}=0 \tag{A13}
\end{equation*}
$$

Thus, equation (A12) at the point of choking becomes

$$
\begin{equation*}
\left.G_{\max }^{2}=-\frac{d v}{d P}\right]_{t}^{-1} \tag{A14}
\end{equation*}
$$

By using the definition (A8), equation (A14) can be written as

$$
\begin{equation*}
G_{\max }^{2}=-\left[x \frac{d v_{g}}{d P}+(1-x) \frac{d v_{l}}{d P}+\left(v_{g}-v_{l}\right) \frac{d x}{d P}\right]_{t}^{-1} \tag{A15}
\end{equation*}
$$

Either equation (A14) or equation (A15) can be solved, together with equation (A11), to determine the choked flow rate and the choking pressure.

The specific volume is, of course, a function of two variables, so a path must be determined. For a nozzle an isentropic path is an appropriate choice. As with specific volume, the entropy for a two-phase medium is defined as a percentage contribution from the liquid component and the vapor component:

$$
\begin{equation*}
\mathrm{S}=\mathrm{xS} \mathrm{~g}_{\mathrm{g}}+(1-\mathrm{x}) \mathrm{S}_{l}=\text { Constant }=\mathrm{S}_{\mathrm{o}} \tag{A16}
\end{equation*}
$$

Since the entropy is constant, equation (A16) defines the quality as long as the system is in thermodynamic equilibrium.

The model of Henry and Fauske (ref. 17) proposes a departure from thermodynamic equilibrium that they feel is consistent with the observations made in various two-phase-choked-flow experiments. The basic equations, (A11) and (A14), remain valid. The modifications are in the definitions of the specific volume, equation (A8), and the volume derivative, equation (A15). The basic statement is that there is insufficient time for any significant change in quality during the acceleration to the throat; however, the rate of quality change would still be significant. Thus, the quality is assumed to be constant at the stagnation value, and equation (A8) becomes

$$
\begin{equation*}
v=x_{o} v_{g}+\left(1-x_{o}\right) v_{l} \tag{A17}
\end{equation*}
$$

and for the present case of subcooled, liquid inlet-stagnation conditions this reduces to

$$
\begin{equation*}
\mathrm{v}=\mathrm{v}_{l} \tag{A18}
\end{equation*}
$$

Henry and Fauske also assumed $\mathrm{dv}_{l} / \mathrm{dP} \approx 0$ and proposed that the nonequilibrium derivative of the quality could be related to equilibrium by

$$
\begin{equation*}
\left.\left.\frac{\mathrm{dx}}{\mathrm{dP}}\right]_{\mathrm{t}}=\mathrm{N} \frac{\mathrm{dx}}{\mathrm{dP}}\right]_{\mathrm{t}} \tag{A19}
\end{equation*}
$$

where

$$
\mathrm{N}= \begin{cases}\frac{\mathrm{x}_{\mathrm{e}}}{0.14} & \text { for } \mathrm{x}_{\mathrm{e}}<0.14 \\ 1.0 & \text { for } \mathrm{x}_{\mathrm{e}} \geq 0.14\end{cases}
$$

Thus, equation (A15) becomes

$$
\begin{equation*}
G_{\max }^{2}=-\left[x_{0} \frac{d v_{g}}{d P}+\left(v_{g}-v_{l}\right) N \frac{d x_{e}}{d P}\right]_{t}^{-1} \tag{A20}
\end{equation*}
$$

which for subcooled inlets reduces to

$$
\begin{equation*}
G_{\max }^{2}=-\left[\left(v_{g}-v_{l}\right) N \frac{d x_{e}}{d P}\right]_{\mathrm{t}}^{-1} \tag{A21}
\end{equation*}
$$

As a computational convenience, Henry and Fauske (ref. 17) differentiated equation (A16) and used the subcooled-inlet condition to express equation (A21) in the following entropy terms:

$$
\begin{equation*}
\mathrm{G}_{\max }^{2}=-\left[\frac{\mathrm{N}\left(\mathrm{v}_{\mathrm{g}}-\mathrm{v}_{l}\right)}{\left(\mathrm{S}_{\mathrm{g}}-\mathrm{S}_{l}\right)} \frac{\mathrm{dS}}{l}{ }_{\mathrm{dP}}\right]_{\mathrm{t}}^{-1} \tag{A22}
\end{equation*}
$$

where $S_{g}$ and $S_{l}$ were assumed to be at the saturation conditions corresponding to $\mathrm{T}_{\mathrm{o}}$. For the present work the stagnation conditions were frequently too far from the saturation locus to allow that assumption, so the Henry-Fauske program was modified to compute $v_{l}, S_{g}$, and $S_{l}$ at the saturation pressure corresponding to an isentropic expansion from the stagnation conditions to the saturation locus. This is designated $P_{\text {sat }}\left(S_{0}\right)$. For the convenience of the reader and to point out the differences, both $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{S}_{\mathrm{o}}\right)$ and $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{T}_{\mathrm{o}}\right)$ are included in the data tables. Because of this modification, we refer to the model as the modified Henry -Fauske model.

## REFERENCES

1. Hsu, Y. Y.: Review of Critical Flow Rate, Propagation of Pressure Pulse, and Sonic Velocity in Two-Phase Media. NASA TN D-6814, 1972.
2. Henry, R. E.; Grolmes, M. A.; and Fauske, H. K.: Pressure Drop and Compressible Flow of Cryogenic Liquid-Vapor Mixtures. Heat Transfer at Low Temperatures, W. Frost, ed., Plenum Press, 1975, pp. 229-259.
3. Smith, R. V.; Randall, K. R.; and Epp, R.: Critical Two-Phase Flow for Cryogenic Fluids. (NBS TN-633, National Bureau Standards; NASA Order W-13300.) NASA CR-130793, 1973.
4. Lahey, R. T., Jr.; and Wallis, G. B., eds.: Nonequilibrium Two-Phase Flows, American Society of Mechanical Engineers, 1975.
5. Simoneau, R. J.; et al.: Two-Phase Critical Discharge of Liquid Nitrogen. Progress in Refrigeration Science and Technology, Proceedings of the 13th International Congress of Refrigeration. Vol. 1, International Institute of Refriger ation by Avi. Publ. Co., Inc., 1973, pp. 293-297.
6. Hendricks, R. C.; Simoneau, R. J.; and Ehlers, R. C.: Choked Flow of Fluid Nitrogen with Emphasis on the Thermodynamic Critical Region. Advances in Cryogenic Engineering, Vol. 18, K. D. Timmerhaus, ed., Plenum Press, 1973, pp. 150-161.
7. Simoneau, R. J.: Pressure Distribution in a Converging-Diverging Nozzle During Two-Phase Choked Flow of Subcooled Nitrogen. Nonequilibrium Two-Phase Flows, R. T. Lahey, Jr., and G. B. Wallis, eds., American Society of Mechanical Engineers, 1975, pp. 37-45.
8. Hendricks, R. C.; Simoneau, R. J.; and Barrows, R. F.: Two-Phase Choked Flow of Subcooled Oxygen and Nitrogen. NASA TN D-8149, 1976.
9. Hendricks, R. C.; Simoneau, R. J.; and Hsu, Y. Y.: A Visual Study of Radial Inward Choked Flow of Liquid Nitrogen. Advances in Cryogenic Engineering, Vol. 20. K. D. Timmerhaus, ed., Plenum Press, 1975, pp. 370-382.
10. Simoneau, R. J.: Two-Phase Choked Flow of Subcooled Nitrogen Through a Slit. Proceedings of the Tenth Southeastern Seminar on Thermal Sciences, R. G. Watts and H. H. Sogin, eds., Tulane Univ., 1974, pp. 225-238.
11. Simoneau, R. J.: Maximum Two-Phase Flow Rates of Subcooled Nitrogen Through a Sharp-Edged Orifice. Advances in Cryogenic Engineering, Vol. 21. K. D. Timmerhaus and D. H. Weitzel, eds., Plenum Press, 1975, pp. 299-306.
12. Hendricks, R. C.; and Simoneau, R. J.: Two-Phase Choked Flow in Tubes with Very Large L/D. Advances in Cryogenic Engineering, Vol. 23. K. D. Timmerhaus, ed., Plenum Press, 1978, pp. 265-275.
13. Sozzi, G. L.; and Sutherland, W. A.: Critical Flow of Saturated and Subcooled Water at High Pressure. Nonequilibrium Two-Phase Flows, R. T. Lahey, Jr., and G. B. Wallis, eds., American Society of Mechanical Engineers, 1975, pp. 19-25.
14. Schrock, V. E.; Starkman, E. S.; and Brown, R. A.: Flashing Flow of Initially Subcooled Water in Convergent-Divergent Nozzles. J. Heat Transfer, vol. 99, no. 2, May 1977, pp. 263-268.
15. Tangren, R. F.; Dodge, C. H.; and Seifert, H. S.: Compressibility Effects in Two-Phase Flow. J. Appl. Phys., vol. 20, no. 7, July 1949, pp. 637-645.
16. Collins, R. L.: Choked Expansion of Subcooled Water and the I.H.E. Flow Model. J. Heat Transfer, vol. 100, no. 2, May 1978, pp. 275-280.
17. Henry, R. E.; and Fauske, H. K.: The Two-Phase Critical Flow of One-Component Mixtures in Nozzles, Orifices, and Short Tubes. J. Heat Transfer, vol. 93, no. 2, May 1971, pp. 179-187.
18. Hendricks, R. C.; and Simoneau, R. J.: Application of the Principle of Corresponding States to Two-Phase Choked Flow. NASA TM X-68193, 1973.
19. Hendricks, R. C.; Normalizing Parameters for the Critical Flow Rate of Simple Fluids Through Nozzles. Proceedings of the Fifth International Cryogenic Engineering Conference, K. Mendelssohn, ed., IPC Science and Technology Press, (England), 1974, pp. 278-281. (Also NASA TM X-71545, 1974.)
20. Hendricks, R. C.; Baron, A. K.; and Peller, I. C.: GASP: A Computer Code for Calculating the Thermodynamic and Transport Properties for Ten Fluids: Parahydrogen, Helium, Neon, Methane, Nitrogen, Carbon Monoxide, Oxygen, Fluorine, Argon, and Carbon Dioxide. NASA TN D-7808, 1975.
21. Bean, Howard S., ed.: Fluid Meters, Their Theory and Application. Sixth ed. American Society of Mechanical Engineers, 1971, p. 216.
22. Mealey, C.; and Kee, L.: Computer-Controlled Central Digital Data Acquisition System. NASA TN D-3904, 1967.
23. Bender, E.: Equations of State Exactly Representing the Phase Behavior of Pure Substances. Proceedings of the Fifth Symposium on Thermophysical Properties. American Society of Mechanical Engineers, 1970, pp. 227-235.

TABLE I. - CONICAL CONVERGING-

## DIVERGING NOZZLE

(a) Dimensions

(b) Tap locations (referenced to throat)

| Station <br> (tap) | Axial dis- <br> tance, <br> X, <br> cm | Radius, <br> r, <br> cm | Ratio of area <br> to throat <br> area, <br> $\pm 0.014 \mathrm{~cm}$ |
| :---: | :---: | :---: | :---: |
| 0 | -9.1 | ----- |  |
| 1 | -5.062 | 0.645 | 13.18 |
| 2 | -3.066 | .408 | 5.26 |
| 3 | -2.263 | .312 | 3.08 |
| 4 | -1.984 | .279 | 2.46 |
| 5 | -1.692 | .244 | 1.88 |
| 6 | -1.052 | .178 | 1.00 |
| 7 | -.536 | .178 | 1.00 |
| 8 | -.185 | .178 | 1.00 |
| 9 | .112 | .185 | 1.08 |
| 10 | .455 | .208 | 1.37 |
| 11 | .940 | .240 | 1.82 |
| 12 | 1.933 | .306 | 2.95 |
| 13 | 7.943 | .703 | 15.61 |
| 14 | 12.939 | 1.033 | 33.73 |
| 15 | 17.943 | 1.363 | 58.79 |
| B | 22.0 | ---- | $\mathrm{b}_{\infty}$ |

${ }^{a^{\prime}}$ Inlet mixing chamber.
${ }^{\mathrm{b}}$ Outlet mixing chamber.

TABLE II. - TWO-DIMENSIONAL CONVERGING-

## DIVERGING NOZZLE

(a) Dimensions

|  |  |
| :---: | :---: |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |
|  |  |

(b) Tap locations (referenced to throat)

| Station (tap) | Axial distance, X , $\mathrm{cm} \pm 0.015 \mathrm{~cm}$ |  | Channel height, cm | Ratio of area to throat area, $A / A_{t}$ |
| :---: | :---: | :---: | :---: | :---: |
|  | Straight wall | Contoured wall |  |  |
| 0 | -21.0 |  | ----- | $\mathrm{b}_{\infty}$ |
| 1 | -15.824 | ------ | 1.899 | 17.34 |
| 2 | -11.824 | ------ | 1.419 | 13.02 |
| 3 | -7.826 | ------ | . 939 | 8.61 |
| 4 | -4.072 |  | . 489 | 4.49 |
| 5 | -2.065 | ------ | . 248 | 2.28 |
| 6 | -1.349 | -1.355 | . 162 | 1.49 |
| 7 | -. 876 | ------ | . 109 | 1.00 |
| 8 | -. 231 | -. 224 | . 109 | 1.00 |
| 9 | . 508 | . 491 | . 135 | 1.24 |
| 10 | 1.392 | ------ | . 181 | 1.66 |
| 11 | 3.828 | ------ | . 308. | 2.83 |
| 12 | 6.370 | ------ | . 441 | 4.05 |
| B | 10.1 | ------ |  | $\mathbf{c}_{\infty}$ |

${ }^{\mathrm{a}}$ Not possible to make tolerance estimate.
$\mathrm{b}_{\text {Inlet mixing chamber }}$.
${ }^{c}$ Outlet mixing chamber.

TABLE III. - ELLIPTICAL CONVERGINGDIVERGING NOZZLE
(a) Dimensions

(b) Tap locations (referenced to throat)

| Station <br> (tap) | Axial dis- <br> tance, <br> X, <br> cm | Radius, <br> r, <br> cm | Ratio of area <br> to throat <br> area, <br> $\mathrm{A} / \mathrm{A}_{\mathrm{t}}$ |
| :---: | :---: | :---: | :---: |
| 0 | -4.605 | $--\cdots-$ | $\mathrm{a}_{\infty}$ |
| 1 | -2.390 | 0.747 | 25.82 |
| 2 | -1.115 | .274 | 3.47 |
| 3 | -.747 | .164 | 1.25 |
| 4 | -.297 | .147 | 1.00 |
| 5 | -.150 | .147 | 1.00 |
| 6 | .373 | .216 | 2.16 |
| 7 | 1.430 | .290 | 3.89 |
| 8 | 3.942 | .467 | 10.09 |
| 9 | 6.452 | .721 | 24.06 |

${ }^{\mathrm{a}}$ Inlet mixing chamber.

TABLE IV. - ERROR ESTIMATES
(a) For parameters measured in 110-liter rig

| Parameter | Range | Percent of aver age value | Absolute |
| :---: | :---: | :---: | :---: |
| Pressure, $\mathrm{N} / \mathrm{cm}^{2}$ : |  |  |  |
| Stagnation, $\mathrm{P}_{\mathrm{o}}$ | 60-680 | $\pm 0.4$ | $\pm 1.4$ |
| Throat, $\mathrm{P}_{\mathrm{t}}$ | 33-275 | $\pm .9$ | ------- |
| Back, $\mathrm{P}_{\mathrm{b}}$ | 20-100 | $\pm 2.3$ | ------- |
| Axial, $\mathbf{P}$ | 30-550 | $\pm .5$ | -------- |
| Temperature, K: |  |  |  |
| Stagnation, $\mathrm{T}_{\mathbf{o}}$ | 90-130 | $\pm .1$ | $\pm .1$ |
| Back, $\mathrm{T}_{\mathrm{b}}$ | 84-104 | $\pm .1$ | $\pm .1$ |
| Mass flux, $G, g / \mathrm{cm}^{2}$. sec: 95-125 K isotherms | 1600-9000 | $\pm 1.4$ | $\pm 75$ |
| 130 K isotherm | 1700-6000 | $\pm 2.1$ | $\pm 80$ |

(b) For parameters measured in 375-liter rig

| Pressure, N/cm ${ }^{2}:$ | - |  |  |
| :--- | :---: | :---: | :---: |
| Nitrogen | $150-950$ | $\pm 0.4$ | $\pm 2.5$ |
| Methane | $150-925$ | $\pm .4$ | $\pm 2.5$ |
| Hydrogen | $130-425$ | $\pm .5$ | $\pm 2.0$ |
| Temperature, K: |  |  |  |
| Nitrogen | $87-235$ | $\pm .15$ | $\pm .25$ |
| Methane | $121-200$ | $\pm .10$ | $\pm .20$ |
| Hydrogen | $27.2-32.3$ | $\pm .25$ | $\pm .10$ |
| Mass flux, g/cm ${ }^{2} \cdot \mathrm{sec}:$ |  |  |  |
| Nitrogen | $200-10600$ | $\pm 2.0$ | $\pm 5-140$ |
| Methane | $1700-7100$ | $\pm 2.5$ | $\pm 100$ |
| Hydrogen | $900-2500$ | $\pm 3.0$ | $\pm 30$ |
|  |  |  |  |

TABLE V. - SUMMARY OF DATA TABLES

${ }^{\mathbf{a}}$ Flow-rate and pressure-ratio data only - no profiles.

TABLE VI. - DATA FOR NITROGEN - $7^{\circ}$ CONICAL NOZZLE
(a) Stagnation temperature, $\mathrm{T}_{0}, 95.0 \mathrm{~K}$

| Reading | Stagna- | Stagna- | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure$\begin{gathered} \mathrm{P}_{\mathrm{b}}{ }^{\prime}{ }^{\mathrm{N} / \mathrm{cm}_{2}} \end{gathered}$ | Temperature at back condition, T , K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max }, \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | Stagnation entropy,$\begin{gathered} \mathrm{S}_{\mathrm{O}}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagna tion entropy,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}\left(\mathrm{~S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation <br> pressure <br> at stagna- <br> tion tem- <br> perature, $\begin{gathered} \mathbf{P}_{\mathrm{sat}}{ }^{\left(\mathrm{T}_{0}\right)}, \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | tion temperature, $\mathrm{T}_{\mathrm{o}}$, K | $\begin{aligned} & \text { tion } \\ & \text { pres- } \\ & \text { sure, } \\ & { }^{P_{0}^{\prime}} \\ & \mathrm{N} / \mathrm{cm}^{2} \end{aligned}$ | $P_{1}$ | $\mathbf{P}_{2}$ | $\mathrm{P}_{3}$ |  | $P_{5}$ | $\mathrm{P}_{6}$ |  |  |  |  | $\mathrm{P}_{11}{ }^{\text {P }}$ |  |  | $\mid P_{14}$ <br> $\mid$ | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 101-562 | 95.0 | 60 | 59 | 59 | 57 | 56 | 531 | $40^{1}$ | 35 | 33 | 26 | 23\| | 22 | 24 | $24{ }^{\prime}$ | 24 | 25 | 25 | 86.2 | 0.543 | 1580 | 0.840 | 54 | 54 |
| 551 | 94.7 | 78 | 76 | -76, | 73\| | 72 | 67 | 47. | 43 | 39। | 30 | 24 | 19 | 18 | 20 | 20\| | 20 | 21 | 84.1 | . 504 | 2000 | . 830 | 52 | 53 |
| 504 | 95.0 | 90 | 89 | 88 | 85 | $82^{\prime}$ | 76 | 49 | 47 | 44 | 34 | 29. | 31 | 32 | 34 | 36 | 37 | 37 | 90.6 | . 491 | 2290 | . 836 | 53 | 54 |
| 559 | 94.9 | 111 | 110 | 109 | 103 | 100 | 90 | 51 | 48 | 48 | 37 | 30 | 23 | 21 | 25 | 25 | 26 | 26 | 86.9 | . 434 | 2840 | . 830 | 52 | 53 |
| 547 | 94.8 | 130 | 129 | 127 | 120 | 115 | 102 | 51 | 47 | 49 | 39 | 32 | 25 | 24 | 30 | 31 | 32 ' | 33 | 88.0 | . 376 | 3240 | . 827 | 52 |  |
| 491 | 95.3 | 142 | 141 | 139 | 131 | 125 | 110 | 53 | 48 | 49 | 40 | 33. | 30 | 38 | 50 | 54 ! | 55 | 55 | 95.2 | . 346 | 3450 | . 837 | 54 |  |
| 544 | 94.8 | 195 | 193 | 189 | 178 | 168 | 145 | 54 | 46 | 50 | 41 | 35 | 28 | 20 | 33 | 36 | 39 | 40 | 90.7 | . 257 | 4350 | . 821 | 51 | $\dagger$ |
| 541 | 95.1 | 246 | 244 | 238 | 223 | 209 | 179 | 56 | 46 | 51 | 42 | 361 | 29 | 35 | 83 | 84 | 85 | 85 | 95.5 | . 207 | 5050 | . 822 | 51 | 54 |
| 500 | 95.0 | 284 | 282 | 275 | ; 257 | 240 | 204 | 58 | 45 | 50 | 42 | $36{ }^{\text { }}$ | 29 | 20 | 43 | 55 | 58 | 58 | 95.5 | . 177 | 5540 | . 815 | 49 | 54 |
| 539 | 95.3 | 338 | 336 | 327 | 304 | 285 | \| 241 | 61 | 46 | 51 | 43 | 37 | 30 | 21 | 52 | 70 | 71 | 72 | 95.9 | . 151 | 6130 | . 817 | 50 | 55 |
| 498 | 95.5 | 407 | 405 | 393 | 366 | 341 | 287 | 60 | 46 | 51 | 43 | 38 | 31 | 21 | 41 | 57 | 63 | 64 | 96.4 | . 125 | 6770 | . 814 | 49 | 56 |
| 529 | 95.2 | 411 | 408 | 397 | 368 | 344 | 288 | 63 | 44 | 50 | 42 | 37 |  | 22 | 34 | 43 | 51 | 53 | 94.8 | . 121 | 6850 | . 808 | 48 | 55 |
| 532 | 94.9 | 473 | 471 | 456 | 423 | 394 | 330 | 64 | 43 | 49 | 41 | 36. |  | 21 | 59 | 71 | 87 | 87 | 95.8 | . 103 | 7440 | . 796 | 46 | 54 |
| 524 | 95.3 | 535 | 532 | 515 | 478 | 445 | 371 | 68 | 44 | 49 | 42 | 37 | $\dagger$ | 22 | 46 | 71 | 77 | 77 | 96.3 | . 092 | 7950 | . 799 | 47 | 55 |
| 556 | 95.7 | 594 | 590 | 572 | 530 | 493 | 412 | 73 | 46 | 51 | 43 | 38 | 32 | 23 | 54 | 88 | 91 | 91 | 96.8 | . 085 | 8400 | . 801 | 47 | 57 |
| 488 | 94.9 | 646 | 642 | 622 | 576 | 535 | 444 | 73 | 42 | 46 | 40 | 35 | 30 | 22 | 22 | 26 | 30 | 32 | 89.1 | . 071 | 8840 | . 781 | 44 | 54 |
| 506 | 94.5 | 659 | 655 | 634 | 587 | 545 | 454 | 74 | 42 | 47 | 40 | 35 | 30 | 20 | 33 | 42 | 55 | 61 | 95.9 | . 071 | 8960 | . 771 | 42 | 52 |

TABLE VI. - Continued.
(b) Stagnation temperature, $\mathrm{T}_{0}, 110.0 \mathrm{~K}$

| Reading | Stagna - <br> tion <br> temper- <br> ature, <br> To, <br> K | Stagnation pressure, $\mathrm{P}_{\mathrm{o}},{ }_{2}$$\mathrm{~N} / \mathrm{cm}^{2}$ | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Back pressure, $P_{b}$, $\mathrm{N} / \mathrm{cm}^{2}$ | Tempera- <br> ture at back condition, $T$, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux, $\mathrm{G}_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | $\begin{aligned} & \text { Stagnation } \\ & \text { entropy, } \\ & \mathrm{S}_{\mathrm{O}}, \\ & \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{aligned}$ | Saturation <br> pressure <br> at stagna- <br> tion en- <br> tropy, $\begin{gathered} P_{\text {sat }}\left(S_{o}\right) \\ N / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}\left(\mathrm{~T}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ | $\mathrm{P}_{10}$ | $\mathrm{P}_{11}$ | $\mathrm{P}_{12}$ | $\mathrm{P}_{13}$ | $\mathrm{P}_{14}$ | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 101-906 | 109.9 | 162 | 161 | 160 | 154 | 149 | 138 | 85 | 78 | 71 | 52 | 42 | 33 | 26 | 24 | 27 | 28 | 29 | 87.5 | 0.441 | 2840 | 1.164 | 145 | 146 |
| 920 | 110.3 | 169 | 167 | 166 | 159 | 155 | 143 | 88 | 80 | 73 | 53 | 42 | 34 | 26 | 23 | 27 | 28 | 28 | 87.1 | . 436 | 2860 | 1.173 | 148 | 149 |
| 480 | 109.6 | 178 | 177 | 176 | 168 | 162 | 148 | 88 | 80 | 73 | 55 | 42 | 34 | 29 | 42 | 45 | 46 | 46 | 93.1 | . 411 | 3120 | 1.154 | 141 | 143 |
| 857 | 109.9 | 200 | 199 | 196 | 188 | 182 | 166 | 99 | 91 | 84 | 61 | 46 | 35 | 30 | 48 | 49 | 51 | 52 | 94.8 | . 423 | 3340 | 1.155 | 141 | 145 |
| 478 | 109.7 | 216 | 215 | 213 | 204 | 196 | 179 | 106 | 97 | 87 | 62 | 45 | 35 | 40 | 55 | 58 | 61 | 61 | 96.7 | . 404 | 3510 | 1.149 | 139 | 144 |
| 863 | 110.3 | 241 | 239 | 236 | 226 | 218 | 200 | 124 | 112 | 101 | 71 | 50 | 36 | 28 | 51 | 54 | 56 | 57 | 96.5 | . 418 | 3660 | 1.158 | 143 | 149 |
| 874 | 109.8 | 251 | 250 | 247 | 236 | 228 | 208 | 128 | 117 | 108 | 77 | 54 | 39 | 26 | 28 | 34 | 36 | 36 | 89.9 | . 428 | 3780 | 1.145 | 138 | 145 |
| 457 | 110.4 | 265 | 264 | 260 | 250 | 240 | 220 | 130 | 123 | 113 | 80 | 56 | 40 | 27 | 44 | 48 | 51 | 51 | 94.4 | . 426 | 3880 | 1.156 | 142 | 151 |
| 865 | 110.5 | 268 | 267 | 263 | 252 | 243 | 221 | 135 | 124 | 115 | 82 | 58 | 41 | 27 | 40 | 45 | 47 | 48 | 93.8 | . 429 | 3940 | 1.156 | 142 | 151 |
| 868 | 109.9 | 283 | 281 | 277 | 264 | 254 | 230 | 133 | 123 | 119 | 86 | 62 | 43 | 28 | 28 | 35 | 37 | 38 | 90.8 | . 421 | 4170 | 1.140 | 136 | 146 |
| 879 | 109.9 | 205 | 294 | 288 | 275 | 264 | 238 | 133 | 123 | 121 | 87 | 63 | 45 | 29 | 56 | 61 | 64 | 65 | 97.7 | . 409 | 4360 | 1.137 | 135 | 146 |
| 889 | 109.8 | 307 | 306 | 300 | 286 | 274 | 246 | 133 | 122 | 122 | 89 | 65 | 46 | 30 | 53 | 60 | 63 | 64 | 97.3 | . 398 | 4530 | 1.135 | 134 | 145 |
| 476 | 110.0 | 319 | 318 | 312 | 297 | 284 | 254 | 135 | 124 | 123 | 91 | 67 | 48 | 32 | 65 | 73 | 76 | 76 | 99.7 | . 388 | 4630 | 1.136 | 134 | 146 |
| 902 | 110.4 | 336 | 335 | 328 | 312 | 298 | 267 | 138 | 127 | 127 | 94 | 69 | 49 | 35 | 82 | 91 | 93 | 93 | 103.0 | . 379 | 4800 | 1.141 | 136 | 150 |
| 461 | 110.1 | 348 | 347 | 340 | 323 | 308 | 274 | 135 | 123 | 126 | 93 | 68 | 48 | 30 | 55 | 62 | 66 | 67 | 98.0 | . 362 | 4980 | 1.134 | 134 | 148 |
| 881 | 110.3 | 367 | 366 | 358 | 340 | 323 | 288 | 139 | 125 | 128 | 96 | 71 | 51 | 32 | 71 | 81 | 84 | 84 | 101.5 | . 347 | 5180 | 1.135 | 134 | 150 |
| 891 | 109.9 | 376 | 374 | 366 | 346 | 330 | 292 | 135 | 120 | 124 | 95 | 71 | 51 | 32 | 44 | 52 | 57 | 58 | 96.1 | . 331 | 5310 | 1.124 | 130 | 146 |
| 907 | 110.5 | 390 | 389 | 380 | 360 | 342 | 303 | 140 | 125 | 129 | 98 | 73 | 52 | 33 | 50 | 59 | 64 | 65 | 97.5 | . 331 | 5400 | 1.134 | 134 | 151 |
| 482 | 110.0 | 391 | 390 | 382 | 361 | 342 | 302 | 137 | 120 | 124 | 96 | 71 | 52 | 34 | 61 | 72 | 76 | 76 | 100.0 | . 317 | 5480 | 1.124 | 130 | 147 |
| 910 | 110.1 | 426 | 424 | 414 | 391 | 371 | 326 | 138 | 120 | 125 | 97 | 73 | 53 | 34 | 35 | 43 | 48 | 50 | 94.0 | . 294 | 5820 | 1.120 | 129 |  |
| 894 | 110.0 | 454 | 452 | 441 | 416 | 393 | 344 | 137 | 117 | 124 | 97 | 74 | 54 | 34 | 36 | 44 | 50 | 52 | 94.4 | . 273 | 6110 | 1.115 | 127 |  |
| 921 | 110.0 | 472 | 470 | 459 | 432 | 409 | 357 | 139 | 118 | 125 | 98 | 75 | 55 | 35 | 66 | 80 | 85 | 86 | 101.5 | . 265 | 6250 | 1.112 | 126 | $\dagger$ |
| 895 | 109.9 | 520 | 518 | 505 | 475 | 448 | 388 | 139 | 115 | 123 | 98 | 75 | 55 | 35 | 78 | 98 | 101 | 101 | 104.2 | . 236 | 6710 | 1.103 | 123 | 146 |
| 914 | 110.1 | 543 | 541 | 527 | 495 | 467 | 404 | 140 | 115 | 124 | 99 | 76 | 56 | 35 | 37 | 47 | 54 | 56 | 95.5 | . 228 | 6910 | 1.103 | 123 | 147 |
| 456 | 109.8 | 566 | 564 | 549 | 515 | 485 | 419 | 138 | 111 | 121 | 96 | 75 | 55 | 34 | 18 | 34 | 34 | 37 | 90.7 | . 214 | 7130 | 1.093 | 120 | 145 |
| 474 | 110.3 | 617 | 614 | 598 | 561 | 527 | 454 | 145 | 114 | 123 | 99 | 76 | 57 | 37 | 48 | 64 | 73 | 74 | 99.5 | . 199 | 7510 | 1.097 | 121 | 150 |
| 918 | 109.9 | 645 | 642 | 625 | 585 | 550 | 472 | 145 | 112 | 122 | 98 | 77 | 58 | 37 | 38 | 49 | 58 | 61 | 96.8 | . 189 | 7710 | 1.086 | 117 | 146 |
| 460 | 109.9 | 664 | 660 | 642 | 601 | 564 | 484 | 142 | 109 | 120 | 97 | 75 | 56 | 35 | 19 | 30 | 36 | 40 | 91.7 | . 181 | 7890 | 1.083 | 116 | 146 |

TABLE VI - Continued
(c) Stagnation temperature, $\mathrm{T}_{0}, 119.3 \mathrm{~K}$

| Reading | Stagna- | Stagna- | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$, |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\begin{gathered} \mathrm{P}_{\mathrm{b}} \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, Tb, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{0}$ | Maximum mass flux, $G_{\text {max }}$, $\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} \mathrm{S}_{0} \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~S}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} P_{s a t}\left(T_{0}\right), \\ N / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | tion temperature, $\mathrm{T}_{\mathrm{o}}$, K | tion <br> pressure, $\begin{gathered} P_{0}, \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | $P_{1}$ | $\mathbf{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $P_{5}$ | $\mathrm{P}_{6}$ | $P_{7}$ | $\mathrm{P}_{8}$ | $P_{9}$ | $\mathrm{P}_{10}{ }^{\mathrm{F}}$ | $\mathrm{P}_{11}$ | $12$ | $P_{13}{ }_{1}$ | $P_{14}$ | $P_{15}$ |  |  |  |  |  |  |  |
| 101-612 | 119.2 | 263 | 262 | 259 | 252 | 246 | 231 | 168 | ' 163 | 160 | 157 | 156 | 129 |  | 31 | 37 | 41 | 43 | 92.4 | 0.608 | 2990 | 1.387 | 236 | 242 |
| 648 | 119.3 | 281 | 280 | 276 | 268 | 260 | 243 | 189 | ${ }_{161}$ | 156 | 152 | 152 | 134: | 69 : | 24 | 29 | 32 | 34 | 89.6 | . 555 | 3270 | 1.381 | 234 | 243 |
| 587 ! | 119.3 | 289 | 288 ' | 285' | ' 275 | 268 | $249{ }^{1}$ | ' 167 | 158 | - 152 | 147 | 148 | 135 | 73 | 58 | 67 | 71 | 71 | 98.9 | . 524 | 3440 | 1.377 | 232 | 243 |
| $669{ }^{\text {' }}$ | 119.1 | 294 | 293 | 289 | 279 | $270{ }^{\text {' }}$ | 250 | ${ }^{162}$ ' | ' 152 | $145{ }^{\prime}$ | '139 | 140 | 132 | 76 ! | 26 | $31^{\prime}$ | 34 | 37 | 90.5 | . 492 | 3570 | 1.368 | 228 | 240 |
| 657 | 119.1 | 313 | 312 | 307 | 295 | 285 | 263 | 164 | 151 | 141 | 133 | 135 | $129{ }^{\prime}$ | 80 | 40 | 47 | 52 | 54 | 95.3 | . 451 | 3790 | 1.362 | 225 | 240 |
| 656 | 118.8 | 317 | 316 | 311 | 299 | 289 | 265 | 162 | 147 | 136 | 124 | 127 | 123 | 83 | 28. | 33 | 37 | 40 | 91.3 | . 428 | 3900 | 1.351 | 221 | 237 |
| 603 | 118.8 | 322 | 321 | 316 | 303 | 293 | 269 | 163 | 148 | 136 | 123 | 126 | 123 | 83 | 28 | 34 | 38 | 40 | 91.5 | . 423 | 3950 | 1.350 | 220 | 237 |
| 606 | 119.2 | 345 | 344 | 339 | 325 | 314 | 288 | 175 | 156 | 141 | 122 | 126 | 123 | 83 | 62 | 73 | 77 | 78 | 100.2 | . 410 | 4110 | 1.351 | 221 | 241 |
| 597 | 119.1 | 361 | 359 | 354 | 340 | 329 | 301 | 183 | 164 | 147 | 111 | 118 | 116 | 85 | 30 | 36 | 40 | 42 | 92.0 | . 407 | 4210 | 1.343 | 217 | 240 |
| 572 | 119.8 | 379 | 378 | 371 | 357 | 344 | 315 | 192 | 172 | 154 | 125 | 129 | 125 | 85 | 55 | 67 | 72 | 73 | 99.2 | . 408 | 4340 | 1.355 | 223 | 249 |
| 615 | 119.7 | 407 | 406 | 399 | 383 | 369 | 338 | 205 | 187 | 169 | 118 | 121 | 119 | 85 | 30 | 37 | 41 | 44 | 92.8 | . 416 | 4540 | 1.345 | 218 | 248 |
| 617 | 119.0 | 441 | 440 | 432 | 413 | 396 | 359 | 204 | 188 | 182 | 126 | 87 | 90 | 82 | 35 | 42 | 47 | 50 | 93.9 | . 412 | 4970 | 1.316 | 206 | 239 |
| 590 | 119.4 | 474 | 472 | 464 | 443 | 425 | 384 | 209 | 192 | 190 | 132 | 92 | 97 | 86 | 58 | 73 | 79 | 80 | 100.5 | . 400 | 5260 | 1.318 | 207 | 244 |
| 649 | 119.9 | 503 | 501 | 491 | 468 | 448 | . 403 | 213 | 194 | 195 | 138. | . 95 | 99 | 88 | 68 | 85 | 91 | 91 | 102.6 | . 387 | 5480 | 1.323 | 209 | 251 |
| 578 | 119.5 | 525 | 523 | 512 | 487 | 466 | 417 | 210 | 189 | 192 | 138 | , 94 | 87 | 81 | 58 | 73 | 80 | 81 | 100.9 | . 366 | 5760 | 1.307 | 202 | 245 |
| 619 | 119.5 | 549 | 547 | 535 | 508 | 485 | 433 | 210 | 187 | 191 | ,139 | 96 | 76 | 74 | 35. | 45 | 52 | 56 | 95.8 | . 348 | 5980 | 1.301 | 200 | 245 |
| 651 | 119.6 | 578 | 576 | 563 | 534 | 509 | 452 | 210 | 186 | '191 | 141 | 97 | 71 | 72 | 55 | 71 | 79 | 81 | 100.8 | . 330 | 6260 | 1.296 | 198 | 246 |
| 594 | 119.8 | 602 | 600 | 587 | 556 | 531 |  | 212 | 186 | 192 | 144 |  | 70 | 71 | 54 | 70 | 79 | 80 | 100.8 | . 319 | 6430 | 1.296 | 198 | 249 |
| 671 | 119.2 | 624 | 621 | 607 | 574 | 546 |  | 206 | 178 | 185 | 141 |  | 68 | 54 | 37 | 48 | 56 | 61 | 96.8 | . 297 | 6690 | 1.278 | 190 | 241 |
| 592 | 119.2 | 661 | 658 | 643 | 607 | 577 | 507 | 208 | 177 | 185 | 142 |  | 70 | 45 | 48 | 66 | 75 | 78 | 100.3 | . 281 | 6970 | 1.272 | 187 | 242 |

TABLE VI. - Continued
(d) Stagnation temperature, $\mathrm{T}_{0}, 124.3 \mathrm{~K}$

| Reading |  |  | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\begin{gathered} \mathrm{P}_{\mathrm{b}^{\prime}} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | $\begin{aligned} & \text { Maximum } \\ & \text { mass flux, } \\ & G_{\max }, \\ & \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{aligned}$ | Stagnation entropy,$\begin{gathered} S_{o^{\prime}} \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}\left(\mathrm{~S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathbf{P}_{\text {sat }}\left(T_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ | ${ }^{10}$ | $\mathrm{P}_{11}$ | $\mathrm{P}_{12}$ | $\mathrm{P}_{13}{ }^{\text {P }}$ | $\mathrm{P}_{14}$ | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 101-690 | 124.1 | 339 | 339 | 336 | 330 | 325 | 313 | 264 | 258 | 258 | 222 | 142 | 94 | 47 | 17. | 23 | 27 | 29 | 87.7 | 0.762 | 2530 | 1.526 | 292 | 308 |
| 692 | 124.1 | 358 | 358 | 355 | 347 | 340 | 324 | 257 | 249 | 248 | 237 | 169 | 113 | 57 | 20 | 26 | 30 | 33 | 89.2 | . 692 | 2980 | 1.508 | 285 |  |
| 627 | 124.1 | 375 | 375 | 371 | 361 | 353 | 334 | 252 | 241 | 239 | 232 | 186 | 125 | 65 | 22 | 29 | 34 | 37 | 90.4 | . 637 | 3310 | 1.492 | 280 |  |
| 694 | 124.1 | 392 | 392 | 387 | 376 | 366 | 343 | 248 | 236 | 232 | 225 | 195 | 133 | 70 | 23 | 30 | 35 | 38 | 90.9 | . 592 | 3590 | 1.483 | 276 | 1 |
| 696 | 124.3 | 409 | 408 | 402 | 390 | 379 | 354 | 248 | 233 | 228 | 221 | 199 | 138 | 74 | 25 | 33 | 38 | 41 | 91.7 | . 557 | 3820 | 1.478 | 274 | 310 |
| 702 | 124.4 | 423 | 421 | 415 | 402 | 390 | 363 | 247 | 231 | 224 | 217 | 201 | 142 | 77 | 53 | 67 | 73 | 73 | 99.4 | . 531 | 4000 | 1.472 | 272 | 312 |
| 628 | 124.4 | 429 | 428 | 422 | 408 | 396 | 368 | 248 | 232 | 224 | 216 | 201 | 144 | 78 | 35 | 46 | 53 | 55 | 95.6 | . 521 | 4080 | 1.469 | 270 | 312 |
| 705 | 124.1 | 437 | 436 | 429 | 414 | 401 | 371 | 242 | 225 | 215 | 206 | 197 | 146 | 80 | 27 | 35 | 41 | 44 | 92.7 | . 491 | 4250 | 1.455 | 265 | 307 |
| 708 | 124.5 | 450 | 449 | 442 | 427 | 413 | 382 | 249 | 231 | 221 | 212 | 201 | 147 | 82 | 47 | 62 | 69 | 70 | 98.8 | . 491 | 4320 | 1.463 | 268 | 314 |
| 712 | 124.2 | 451 | 450 | 443 | 427 | 413 | 381 | 244 | 226 | 215 | 205 | 197 | 148 | 83 | 50 | 64 | 70 | 71 | 99.1 | . 476 | 4400 | 1.452 | 263 | 309 |
| 673 | 124.0 | 456 | 455 | 447 | 431 | 416 | 383 | 242 | 223 | 210 | 197 | 192 | 148 | 84 | 28 | 35 | 41 | 45 | 93.0 | . 460 | 4460 | 1.445 | 261 | 306 |
| 714 | 124.3 | 457 | 456 | 449 | 432 | 417 | 385 | 245 | 227 | 214 | 203 | 196 | 149 | 83 | 39 | 51 | 58 | 61 | 96.9 | . 469 | 4430 | 1. 453 | 264 | 310 |
| 719 | 124.3 | 473 | 472 | 464 | 447 | 431 | 396 | 247 | 228 | 214 | 198 | 193 | 150 | 86 | 45 | $60^{\prime}$ | 68 | 69 | 98.7 | . 452 | 4610 | 1. 446 | 261 | 311 |
| 722 | 124.1 | 483 | 482 | 474 | 455 | 439 | 402 | 246 | 227 | 212 | 190 | 188 | 152 | 87 | 29 | 38 | 45 | 48 | 93.8 | . 440 | 4730 | 1. 436 | 257 | 307 |
| 631 | 124.4 | 492 | 491 | 483 | 464 | 447 | 410 | 250 | 231 | 217 | 193 | 190 | 152 | 88 | 29 | 38 | 45 | 49 | 93.9 | . 440 | 4780 | 1.441 | 259 | 312 |
| 726 | 124.5 | 511 | 510 | 501 | 481 | 463 | 423 | 252 | 234 | 222 | 190 | 189 | 154 | 89 | 30 | 39 | 46 | 50 | 94.3 | . 435 | 4950 | 1. 437 | 258 | 314 |
| 677 | 124.3 | 530 | 528 | 519 | 497 | 478 | 435 | 250 | 231 | 223 | 179 | 180 | 157 | 91 | 30 | 39 | 47 | 50 | 94.6 | . 421 | 5170 | 1.425 | 252 | 311 |
| 633 | 124.7 | 562 | 560 | 549 | 525 | 504 | 457 | 254 | 233 | 230 | 178 | 181 | 160 | 93 | 31 | 42 | 50 | 54 | 95.4 | . 410 | 5430 | 1.425 | 252 | 317 |
| 733 | 124.5 | 584 | 582 | 571 | 545 | 522 | 471 | 251 | 229 | 229 | 167 | 170 | 159 | 95 | 51 | 68 | 77 | 79 | 100.5 | . 392 | 5680 | 1.410 | 246 | 313 |
| 678 | 124.5 | 598 | 596 | 584 | 557 | 533 | 480 | 249 | 226 | 228 | 165 | 167 | 158 | 95 | 55 | 73 | 83 | 84 | 101.3 | . 382 | 5810 | 1.406 | 244 | 313 |
| 636 | 124.8 | 632 | 630 | 617 | 588 | 562 | 505 | 254 | 228 | 231 | 168 | 167 | 158 | 98 | 86 | 76 | -86 | 88 | 102.1 | . 366 | 6070 | 1.405 | 244 | 319 |
| 680 | 124.9 | 666 | 664 | 650 | 618 | 590 | 527 | 253 | 225 | 229 | 167 | 158 | 154 | 99 | 55 | 75 | - 85 | 86 | 102.0 | . 344 | 6320 | 1. 397 | 240 | 319 |

TABLE VI. - Concluded.
(e) Stagnation temperature, $\mathrm{T}_{0}, 130.0 \mathrm{~K}$

| Reading | Stagna- |  | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\underset{\mathrm{N} / \mathrm{cm}^{\prime}}{ }$ | Temperature at back condition, Tb, K | Ratio of throat pressure to stagnation pressure,$P_{t} / P_{0}$ | Maximum mass flux, $\mathrm{G}_{\text {max }}$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} S_{0^{\prime}} \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}{ }^{\left(\mathrm{T}_{\mathrm{o}}\right),} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | tion temperature, To, K | tion <br> pres- <br> sure, <br> $P_{0}$, <br> $\mathrm{N} / \mathrm{cm}^{2}$ | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $P_{9}$ | $\mathrm{P}_{10} \mathrm{P}$ | $\mathrm{P}_{11}$ | $\mathrm{P}_{12} \mathrm{P}$ | $\mathrm{P}_{13} \mathrm{P}^{\text {a }}$ | $\mathrm{P}_{14} \mathrm{P}$ | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 101-762 | 129.9 | 377 | 376 | 374 | 368 | 363 | 350 | 262 | 244 | 231 | 170 | 111 | 72 | 34 | 7 | 16 | 21 | 23 | 85.5 | 0.613 | 1740 | 2.008 | 322 | (a) |
| 766 | 129.9 | 393 | 393 | 390 | 384 | 379 | 367 | 279 | 260 | 246 | 181 | 117 | 75 | 36 | 9 | 19 | 24 | 26 | 86.3 | . 626 | 1950 | 1.891 | 339 |  |
| 768 | 130.0 | 411 | 411 | 408 | 402 | 396 | 383 | 294 | 269 | 256 | 196 | 133 | 87 | 42 | 11 | 21 | 26 | 28 | 87.5 | . 622 | 2250 | 1.795 | 341 |  |
| 746 | 130.3 | 418 | 418 | 415 | 408 | 402 | 389 | 302 | 274 | 261 | 201 | 137 | 89 | 44 | 42 | 56 | 59 | 59 | 96.3 | . 624 | 2300 | 1.791 | 341 |  |
| 773 | 130.0 | 425 | 424 | 421 | 414 | 407 | 393 | 315 | 286 | 276 | 215 | 5147 | 96 | 47 | 39 | 53 | 57 | 58 | 96.2 | . 651 | 2480 | 1.737 | 338 |  |
| 781 | 130.0 | 427 | 427 | 424 | 417 | 410 | 395 | 324 | 289 | 279 | 217 | 7148 | 98 | 48 | 13 | 24 | 30 | 32 | 88.4 | . 653 | 2520 | 1.733 | 338 |  |
| 778 | 130.0 | 431 | 431 | 427 | 420 | 413 | 398 | 330 | 304 | 282 | 220 | 151 | 100 | 49 | 35 | 49 | 54 | 54 | 95.4 | . 654 | 2590 | 1.721 | 337 |  |
| 805 | 129.9 | 434 | 434 | 431 | 423 | 416 | 400 | 332 | 325 | 317 | 222 | 154 | 102 | 51 | 21 | 35 | 41 | 43 | 92.2 | . 729 | 2650 | 1.704 | 335 |  |
| 806 | 129.9 |  | 434 | 430 | 423 |  |  | 332 | 325 | 308 | 221 | 154 | 101 | 50 | 13 | 23 | 30 | 32 | 88.7 | . 709 | 2630 | 1.708 | 336 |  |
| 789 | 129.9 |  | 434 | 431 | 423 |  |  | 331 | 324 | 319 | 223 | 155 | 102 | 51 | 49 | 64 | 67 | 67 | 98.2 | . 734 | 2660 | 1.702 | 335 |  |
| 790 | 129.9 |  | 433 | 430 | 422 |  |  | 332 | 325 | 310 | 221 | 154 | 101 | 50 | 13 | 24 | 30 | 32 | 88.8 | . 715 | 2640 | 1.708 | 336 |  |
| 791 | 130.1 | $\dagger$ | 433 | 430 | 422 | 1 | 1 | 332 | 324 | 289 | 220 | 153 | 101 | 50 | 25 | 39 | 44 | 45 | 93.0 | . 668 | 2600 | 1.719 | 337 |  |
| 787 | 130.0 | 438 | 438 | 434 | 426 | 419 | 404 | 333 | 326 | 323 | 226 | 157 | 104 | 51 | 72 | 85 | 86 | 87 | 101.8 | . 737 | 2670 | 1.703 | 335 |  |
| 796 | 129.9 | 440 | 440 | 437 | 428 | 421 | 405 | 332 | 324 | 325 | 2331 | \| 161 | 107 | 53 | 13 | 24 | 31 | 33 | 89.1 | . 737 | 2750 | 1.691 | 334 |  |
| 793 | 130.0 | 445 | 445 | 442 | 433 | 425 | 408 | 331 | 323 | 324. | - B | -166 | 110 | 55 | 60 | 75 | 78 | 78 | 100.2 | . 727 | 2850 | 1.684 | 333 |  |
| 799 | 130.1 | 451 | 451 | 447 | 437 | 430 | 411 | 332 | 323 | 324 | 250 | 169 | 113 | 56 | 66 | 81 | 83 | 83 | 101.5 | . 719 | 2880 | 1.682 | 332 |  |
| 802 | 129.9 | 458 | 458 | 454 | 443 | 435 | 414 | 326 | 317 | 317 | 271 | 180 | 121 | 61 | 15 | 26 | 33 | 36 | 90.2 | . 692 | 3100 | 1.658 | 328 |  |
| 845 | 129.9 | 465 | 464 | 459 | 449 | 439 | 418 | 324 | 313 | 314 | 278 | 8185 | 127 | 63 | 30 | 45 | 51 | 53 | 94.9 | . 676 | 3230 | 1.647 | 326 |  |
| 834 | 130.0 | 472 | 471 | 465 | 455 | 445 | 423 | 324 | 314 | 314 | 281 | 189 | 128 | 64 | 17 | 29 | 36 | 39 | 91.1 | . 667 | 3310 | 1.645 | 326 |  |
| 813 | 130.1 | 481 | \| 481 | 475 | 464 | 453 | 429 | 324 | 312 | 313 | 284 | 4194 | 133 | 67 | 17 | 30 | 37 | 40 | 91.4 | . 650 | 3440 | 1.639 | 325 |  |
| 814 | 130.1 | 498 | 498 | 491 | 478 | 466 | 439 | 320 | 307 | 307 | 285. | 5, 203 | 141 | 72 | 42 | 60 | 66 | 67 | 98.2 | . 616 | 3720 | 1.621 | 321 |  |
| 819 | 130.2 | 517 | 517 | 509 | 495 | 481 | 450 | 317 | 302 | 302 | 283 | 311 | 148 | 77 | 20 | 33 | 42 | 44 | 92.6 | . 584 | 3940 | 1.607 | 317 |  |
| 755 | 130.0 | 531 | 530 | 522 | 506 | 491 | 458 | \| 312 | 296 | 295 | 278 | 817 | 153 | 81 | 21 | 34 | 43 | 46 | 93.0 | . 557 | 4190 | 1.590 | 313 |  |
| 821 | 130.0 | 547 | 547 | 538 | 521 | 504 | 468 | 309 | 291 | 290 | 273 | 322 | 157 | 85 | 22 | 35 | 44 | 48 | 93.6 | . 531 | 4390 | 1.578 | 309 |  |
| 848 | 129.7 | 562 | 560 | 551 | 532 | 515 | 476 | 304 | 285 | 283 | 266 | 6226 | 161 | 88 | 24 | 37 | 46 | 50 | 94.1 | . 504 | 4630 | 1.558 | 303 |  |
| 826 | 130.3 | 586 | 585 | 575 | 555 | 536 | 494 | 308 | 288 | 287 | 268 | 8230 | 165 | 91 | 66 | 89 | 94 | 95 | 103.0 | . 490 | 4810 | 1.563 | 304 |  |
| 851 | 129.9 | 597 | 595 | 585 | 563 | 544 | 500 | 302 | 281 | 279 | 259 | 230 | 166 | 93 | 51 | 73 | 81 | 83 | 101.0 | . 468 | 4980 | 1.545 | 298 |  |
| 835 | 130.3 | 637 | 635 | 623 | 598 | 577 | 527 | 305 | 280 | 280 | 255 | 5232 | 170 | 97 | 64 | 88 | 96 | 97 | 103.4 | . 439 | 5320 | 1.537 | 296 |  |
| 751 | 130.0 | 658 | 656 | 644 | 617 | 594 | 540 | 300 | 274 | 275 | 246 | 230 | 170 | 99 | 27 | 43 | 54 | 58 | 96.0 | . 418 | 5570 | 1.521 | 290 |  |
| 852 | 130.1 | 681 | 679 | 666 | 637 | 613 | 556 | 302 | 273 | 275 | 243 | 229 | 171 | 101 | 51 | 76 | 86 | 87 | 102.0 | . 404 | 5750 | 1.514 | 288 |  |
| 717 | 273.0 | 356 | 356 | 353 | 345 | 339 | 323 | 218 | 198 | 186 | 129 | 77 | 45 | 21 | 96 | 97 | 98 | 98 | 266.9 | . 522 | 843 | 3.239 | (a) |  |
| 718 | 271.2 | 356 | 356 | 353 | 346 | 339 | 323 | 218 | 199 | 186 |  | 77 | 45 | 21 | 7 | 12 | 16 | 18 | 263.4 | . 522 | 850 | 3.232 | (a) | $\dagger$ |

${ }^{\mathbf{a}}$ Not applicable.

TABLE VII. - DATA FOR NITROGEN - $3.5^{\circ}$ CONICAL NOZZLE
(a) Stagnation temperature, $T_{0}, 110.1 \mathrm{~K}$

| Reading | Stagna - <br> tion temper -ature,$\mathrm{T}_{\mathrm{o}}$, K | Stagna- <br> tion <br> pres- <br> sure, <br> $P_{0}$, <br> $\mathrm{N} / \mathrm{cm}^{2}$ | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure, $P_{b}$,$\mathrm{N} / \mathrm{cm}^{2}$ | Temperature at back condition, Tb, K | Ratio of throat pressure to stagnation pressure,$P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max } \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | $\begin{gathered} \text { Stagnation } \\ \text { entropy, } \\ \mathrm{S}_{0}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} P_{s a t}\left(S_{0}\right), \\ N / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}{ }^{\left(\mathrm{T}_{0}\right),} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $P_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $P_{6}$ | $\mathrm{P}_{7}$ | $P_{8}$ | $P_{9}$ | $P_{10}$ | $\mathrm{P}_{11} \mathrm{~F}$ | $\mathrm{P}_{12}$ | $P_{13}$ |  | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 101-1425 | 109.8 | 150 | 150 | 150 | 150 | 144 | 134 | 123 | 107 | 91 | 86 | 75 | 28 | 24 | 20 | 17 | 24 | 26 | 86.1 | 0.503 | 2370 | 1.165 | 145 | 145 |
| 1465 | 110.0 | 175 | 175 | 176 | 176 | 166 | 153 | 137 | 116 | 98 | 94 | 82 | 31 | 25 | 21. | 24 | 26 | 28 | 87.6 | . 470 | 2830 | 1.164 | 145 | 146 |
| 1461 | 110.2 | 184 | 184 | 185 | 185 | 175 | 161 | 144 | 123 | 106 | 101 | 89 | 77 | 78 | 80 | 82 | 85 | 88 | 102.0 | . 483 | 2890 | 1.166 | 146 | 148 |
| 1457 | 110.1 | 185 | 185 | 186 | 186 | 176 | 162 | 145 | 1124 | 107 | , 101 | 89 | 32 | 27! | ! 25 | 31 | 33 | 35 | 88.7 | . 479 | 2890 | 1.164 | 145 | 147 |
| 1420 | 110.0 | 199 | 199 | 199 | 199 | 188 | 172 | 153 | 131 | 112 | 105 | 91 | 31 | 25 | 21 | 28 | 30 | 32 | 88.6 | . 459 | 3090 | 1.160 | 143 | 147 |
| 1476 | 110.0 | 207 | 207 | 207 | 207 | 196 | 179 | 158 | 135 | 116 | 108 | 94 | 32 | 25 | 20 | 27. | 29 | 30 | 88.8 | . 453 | 3170 | 1.158 | 142 | 147 |
| 1473 | 110.4 | 214 | 213 | 214 | 214 | 202 | 185 | 163 | 139 | 119 | 111 | 96 | 32 | 26 | 20 | 27 | 29 | 31 | 88.5 | . 449 | 3250 | 1.165 | 145 | 150 |
| 1409 | 109.8 | 223 | 223 | 223 | 223 | 211 | . 192 | 169 | 143 ! | ! 124 | 116 | 100 | 33. | . 26 | 22 | 30 | 32 | 33 | 89.4 | . 447 | 3390 | 1.150 | 139 | 145 |
| 1481 | 110.0 | 234 | 233 | 233 | 234 | 221 | 201 | 175 | 147 | 127 | 119 | 104 | 34 | 27 | 21 | 29 | 31 | 33 | 89.4 | . 445 | 3500 | 1.152 | 140 | 147 |
| 1402 | 110.1 | 244 | 244 | 244 | 244 | 230 | 208 | 182 | 151 | 131 | 124 | 109 | 36 | 28 | 22 | 31 | 32 | 35 | 89.7 | . 446 | 3640 | 1.148 | 139 | 146 |
| 1432 | 109.8 | 269 | 269 | 269 | 269 | 252 | 225 | 194 | 157 | 134 | 126 | 116 | 39 | 30 | 24 | 33 | $35{ }^{\prime}$ | 38 | 90.6 | . 433 | 4000 | 1.140 | 136 | 145 |
| 1398 | 110.2 | 283 | 282 | 282 | 282 | 264 | 235 | 202 | 162 | 138 | 129 | 121 | 41 | 35 | 45 | 52 | 57 | 60 | 96.4 | . 428 | 4180 | 1.147 | 138 | 148 |
| 1375 | 110.1 | 333 | 332 | 333 | 332 | 308 | 271 | 225 | 173 | 140 | 130 | 125 | 44 | $34^{\prime}$ | ' 27 | 40 | 43 | 46 | 92.6 | . 376 | 4760 | 1.137 | 135 | 148 |
| 1376 | 110.1 | 380 | 380 | 380 | 379 | 349 | 303 | 247 | 182 | 141 | 128 | 125 | 46 | 36 | 33 | 52 | 57 | 59 | 96.6 | . 329 | 5320 | 1.128 | 131 | 147 |
| 1387 | 110.2 | 442 | 442 | 442 | 441 | 403 | 346 | 277 | 196 | 145 | 128 | 126 | 83 | 87 | 87 | 88 | 90 | 95 | 103.0 | . 285 | 5950 | 1.120 | 129 | 148 |
| 1379 | 110.1 | 507 | 508 | 507 | 505 | 460 | 391 | 307 | 210 | 148 | 126 | 124 | 50 | 39 | 40 | 62 | 71 | 78 | 100.1 | . 244 | 6580 | 1.108 | 125 | 148 |
| 1391 | 110.1 | 518 | 519 | 518 | 517 | 470 | 399 | 313 | 212 | 148 | 126 | 123 | 50 | 40 | 31 | 46 | 52 | --- | 95.5 | . 238 | 6670 | 1.106 | 124 | 147 |
| 1393 | 110.0 | 579 | 580 | 579 | . 577 | 524 | 440 | 341 | 224 | 151 | 125 | 123 | 51 | 40 | 32 | 47 | 54 | --- | 96.4 | . 212 | 7190 | 1.097 | 121 | 147 |
| 1451 | 110.2 | 598 | 599 | 598 | 596 | 541 | 454 | 351 | 229 | 153 | 126 | 124 | 51 | 41 | 32 | 46 | 53 | 58 | 96.1 | . 207 | 7330 | 1.097 | 121 | 148 |
| 1447 | 109.7 | 654 | 655 | 654 | 651 | 590 | 491 | 377 | 240 | 155 | 124 | 122 | 51 | 41 | 32 | 54 | 62 | 67 | 98.0 | . 186 | 7820 | 1.080 | 115 | 144 |
| 1452 | 110.4 | 666 | - 668 | 667 | 664 | 601 | 501 | 386 | 247 | 160 | 129 | 126 | 52 | 42 | 35 | 65 | 76 | 82 | 100.8 | . 189 | 7850 | 1.091 | 119 | 150 |

TABLE VII. - Concluded.
(b) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 119.3 \mathrm{~K}$

| Reading | Stagna- | Stagna- | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\begin{gathered} \mathrm{P}_{\mathrm{b}},{ }^{2} \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{0}$ | Maximum mass flux, $G_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} \mathrm{S}_{0}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy, $\mathrm{P}_{\text {sat }}\left(\mathrm{S}_{\mathrm{o}}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ | Saturation pressure at stagnation temperature, $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{T}_{\mathrm{o}}\right)$,$\mathrm{N} / \mathrm{cm}^{2}$$\mathrm{N} / \mathrm{cm}^{2}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | tion <br> temper ature, $\begin{gathered} \mathrm{T}_{\mathrm{o}} \\ \mathrm{~K} \end{gathered}$ | tion <br> pres- <br> sure, $\begin{gathered} \mathrm{P}_{\mathrm{o}^{\prime}} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | $\mathrm{P}_{1}$ |  |  |  |  | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ |  | $\mathrm{P}_{10}$ |  |  |  | $\mathrm{P}_{14}$ |  |  |  |  |  |  |  |  |
| 101-1484 | 119.1 | 246 | 246 | 246 | 246 | 238 | 225 | 208 | 187 | 171 | 167 | 163 | 114 | 75 | 52 | 29 | 41 | 46 | 93.1 | 0.661 | 2600 | 1.393 | 239 | 241 |
| 1552 | 119.3 | 254 | 253 | 254 | 254 | 244 | 232 | 211 | 187 | 168 | 164 | 159 | 120 | 81 | 56 | 25 | 29 | 41 | 89.5 | . 628 | 2730 | 1.394 | 239 | 242 |
| 1546 | 119.2 | 284 | 283 | 283 | 284 | 270 | 252 | 225 | 193 | 167 | 160 | 149 | 123 | 102 | 68 | 31 | 30 | 35 | 89.7 | . 524 | 3270 | 1.377 | 232 | 242 |
| 1491 | 119.2 | 298 | 297 | 297 | 297 | 283 | 260 | 233 | 200 | 173 | 164 | 150 | 164 | 167 | 170 | 177 | 188 | 193 | 115.1 | . 504 | 3420 | 1.370 | 229 | 241 |
| 1506 | 119.2 | 298 | 298 | 298 | 298 | 283 | 262 | 233 | 199 | 171 | 162 | 146 | 120 | 105 | 73 | 33 | 47 | 54 | 95.1 | . 491 | 3440 | 1.370 | 229 | 241 |
| 1564 | 119.3 | 317 | 317 | 317 | 317 | 300 | 278 | 245 | 206 | 177 | 166 | 147 | 116 | 107 | 78 | 38 | 58 | 68 | 98.2 | . 464 | 3650 | 1.366 | 227 | 243 |
| 1493 | 119.4 | 327 | 326 | 326 | 326 | 309 | 282 | 250 | 212 | 181 | 169 | 149 | 114 | 107 | 80 | 64 | 76 | 86 | 101.6 | . 455 | 3740 | 1.365 | 227 | 244 |
| 1576 | 119.2 | 344 | 343 | 343 | 343 | 324 | 301 | 263 | 223 | 191 | 178 | 156 | 104 | 101 | 84 | 67 | 80 | 90 | 102.4 | . 454 | 3740 | 1.353 | 222 | 242 |
| 1534 | 119.3 | 347 | 346 | 346 | 346 | 327 | 298 | 262 | 220 | 188 | 175 | 152 | 105 | 102 | 85 | 39 | 32 | 39 | 91.2 | . 438 | 3920 | 1.353 | 222 | 242 |
| 1496 | 119.3 | 350 | 349 | 349 | 348 | 329 | 299 | 264 | 221 | 190 | 177 | 153 | 106 | 103 | 85 | 44 | 68 | 77 | 100.0 | . 437 | 3960 | 1.353 | 222 | 243 |
| 1555 | 119.4 | 350 | 350 | 350 | 350 | 330 | 303 | 265 | 222 | 190 | 177 | 154 | 107 | 104 | 85 | 39 | 37 | 44 | 92.9 | . 439 | 3960 | 1.356 | 223 | 244 |
| 1531 | 119.3 | 366 | 365 | 365 | 365 | 345 | 313 | 274 | 229 | 196 | 183 | 159 | 101 | 100 | 87 | 40 | 33 | 40 | 91.7 | .435 | 4110 | 1.348 | 219 | 243 |
| 1512 | 119.3 | 370 | 370 | 370 | 369 | 348 | 317 | 277 | 231 | 198 | 185 | 161 | 100 | 99 | 87 | 41 | 34 | 41 | 91.8 | . 435 | 4160 | 1.347 | 219 | 243 |
| 1557 | 119.2 | 395 | 395 | 395 | 394 | 370 | 336 | 290 | 239 | 205 | 193 | 171 | 90 | 92 | 85 | 44 | 41 | 49 | 93.8 | . 434 | 4430 | 1.335 | 214 | 241 |
| 1540 | 119.3 | 403 | 403 | 402 | 402 | 378 | 339 | 295 | 240 | 206 | 195 | 174 | 89 | 91 | 85 | 45 | 38 | 47 | 93.7 | . 432 | 4490 | 1.336 | 214 | 243 |
| 1566 | 119.3 | 433 | 433 | 433 | 432 | 404 | 364 | 310 | 246 | 209 | 196 | 182 | 78 | 84 | 81 | 49 | 43 | 53 | 94.6 | . 419 | 4850 | 1.326 | 210 | 243 |
| 1578 | 119.1 | 433 | 433 | 432 | 432 | 403 | 365 | 310 | 247 | 208 | 194 | 185 | 73 | 80 | 77 | 51 | 69 | 80 | 100.7 | . 427 | 4710 | 1.321 | 208 | 240 |
| 1503 | 119.5 | 447 | 447 | 446 | 446 | 417 | 373 | 317 | 253 | 212 | 199 | 186 | 80 | 86 | 82 | 50 | 38 | 47 | 93.3 | . 416 | 4950 | 1.327 | 211 | 245 |
| 1587 | 119.1 | 509 | 510 | 509 | 508 | 471 | 416 | 346 | 263 | 210 | 193 | 187 | 57 | 65 | 65 | 55 | 72 | 86 | 101.5 | . 367 | 5650 | 1.300 | 199 | 240 |
| 1589 | 119.3 | 553 | 553 | 553 | 551 | 510 | 444 | 367 | 274. | . 213 | 194 | 189 | 123 | 137 | 146 | 163 | 191 | 206 | 116.3 | . 342 | 6040 | 1.296 | 198 | 243 |
| 1584 | 119.2 | 564 | 564 | 563 | 562 | 517 | 453 | 371 | 271 | 213 | 193 | 192 | 59 | 53 | 56 | 51 | 70 | 86 | 101.3 | . 341 | 5910 | 1.292 | 196 | 242 |
| 1591 | 119.1 | 580 | 580 | 580 | 578 | 534 | 462 | 379 | 278 | \| 212 | 191 | 187 | 84 | 88 | 90 | 94 | 96 | 105 | 104.5 | . 322 | 6280 | 1.285 | 193 | 240 |
| 1593 | 119.2 | 631 | 632 | 632 | 629 | 579 | 497 | 404 | 290 | 215 | 191 | 187 | 121 | 126 | 126 | 127 | 130 | 140 | 109.4 | . 296 | 6720 | 1.277 | 190 | 242 |
| 1383 | 277.6 | 350 | 350 | 350 | 350 | 339 | 321 | 296 | 257. | 7. 217 | 206 | 188 | 42 | 31 | 42 | 52 | 63 | 73 | 272.5 | . 537 | 817 | 3.263 | (a) | (a) |
| 1384 | 275.4 | 351 | 350 | 350 | 350 | 339 | 321 | 296 | 257 | 7217 | 206 | 188 | 42 | 31 | 20 | 8 | 13 | 18 | 261.8 | . 537 | 820 | 3.254 | (a) | (a) |

${ }^{2}$ Not applicable.

TABLE VIII. - DATA FOR NITROGEN - TWO-DIMENSIONAL NOZZLE
(a) Stagnation temperature, $\mathrm{T}_{0}, 95.3 \mathrm{~K}$

| Reading | Stagnation temper ature, To, K | $\left.\begin{gathered} \text { Stagna- } \\ \text { tion } \\ \text { pres- } \\ \text { sure, } \\ P_{0}^{\prime} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered} \right\rvert\,$ | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | $\begin{gathered} \text { Back- } \\ \text { pressure, } \\ \mathbf{P}_{b^{\prime}}, \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Tempera- <br> ture at back condition, Tb, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux, $\mathrm{G}_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} S_{o^{\prime}}^{\prime} \\ J / g^{\prime} \cdot K \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}\left(\mathrm{~S}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\mathrm{P}_{\text {sat }}\left(\mathrm{T}_{0}\right),$$\mathrm{N} / \mathrm{cm}^{2}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9} \mathrm{P}_{1}$ | $\mathrm{P}_{10} \mathrm{P}_{1}$ | $P_{11} \mid P_{1}$ | $\mathrm{P}_{12} \mathrm{P}^{1}$ | $\mathrm{P}_{13}$ | $\mathrm{P}_{14}$ | $\mathrm{P}_{15}$ |  |  |  |  |  |  |  |
| 1-386 | 94.8 | 71 | 71 | 72 | 71 | 71 | 68 | 62 | 52 | 45 | 38 | 39 | 40 | 41 | 61 | 45 | 38 | 42 | 91.8 | 0.626 | 1730 | 0.833 | 53 | 53 |
| 299 | 95.2 | 108 | 108 | 109 | 108 | 106 | 99 | 86 | 60 | 51 | 46 | 51 | 56 | 63 | 85 | 51 | 45 | 71 | 95.4 | . 468 | 2710 | . 839 | 54 | 55 |
| 114 | 95.4 | 150 | 149 | 150 | 149 | 146 | 133 | 111 | 69 | 55 | 43 | 41 | 51 | 54 | 112 | 55 | 44 | 67 | 96.0 | . 366 | 3530 | . 838 | 54. | 56 |
| 115 | 95.3 | 185 | 184 | 185 | 183 | 179 | 162 | 133 | 77 | 57 | 44 | 41 | 42 | $49^{\prime}$ | 134 | 57 | 44 | 58 | 95.7 | . 309 | 4100 | . 832 | 53 | 55 |
| 117 | 95.6 | 217 | 216 | 216 | 215 | 210 | 189 | 153 | 84 | 60 | 44 | 42 | 41 | 50 | 154 | 60 | 45 | 61 | 96.1 | . 276 | 4530 | . 835 | 53 | 57 |
| 291 | 95.6 | 248 | 247 | 247 | 245 | 239 | 215 | 172 | 90 | 62 | 44 | 44 | 65 | 66 | 174 | 62 | 45 | 102 | 96.1 | . 251 | 4920 | . 833 | 53 | 57 |
| 119 | 95.1 | 287 | 286 | 286 | 284 | 276 | 246 | 196 | 97 | 65 | 41 | 42 | 43 | 58 | 197 | 64 | 43 | 90 | 95.7 | . 225 | 5430 | . 818 | 50 | 55 |
| 111 | 95.3 | 341 | 340 | 340 | 338 | 329 | 293 | 230 | 112 | 70 | 43 | 45 | 45 | 61 | 233 | 71 | 44 | 89 | 97.4 | . 205 | 6010 | . 815 | 50 | 55 |
| 290 | 95.2 | 373 | 372 | 372 | 369 | 359 | 319 | 249 | 117 | 73 | 41 | $42^{\prime}$ | 36 | 56 | 252 | 72 | 42 | 86 | 95.9 | . 196 | 6340 | . 812 | 49 | 55 |
| 121 | 95.8 | 417 | 415 | 415 | 412 | 400 | 355 | 276 | 126 | 78 | 42 | 43. | 35: | 54 | 280 | 77 | 43 | 82 | 96.6 | . 188 | 6730 | . 819 | 50 | 57 |
| ${ }^{\text {a }} 136$ | 95.5 | 462 | 461 | 460 | 457 | 444 | 393 | 304 | 135 | 80 | 41 | 41 | 41 | 69 | 309 | 80 | 42 | 130 | 96.5 | . 174 | 7150 | . 809 | 48 | 56 |
| ${ }^{\text {a }} 137$ | 95.1 | 524 | 522 | 521 | 518 | 502 | 443 | 342 | 147 | 85 | 39 | $39^{\prime}$ | 36 | 63 | 347 | 83 | 39 | 127 | 96.1 | . 161 | 7690 | . 794 | 46 | 54 |
| ${ }^{1} 141$ | 95.4 | 526 | 524 | 522 | 519 | 503 | 445 | 343 | 147 | 84 | 40 | 40 | 35 | 58 | 348 | 84 | 40 | 121 | 96.4 | . 161 | 7700 | . 800 | 47 | 56 |
| 125 | 95.6 | 555 | 553 | 551 | 548 | 533 | 480 | $\mathrm{b}_{208}$ | 162 | 90 | 45 | 42 | 37 | 39 b | ${ }^{\text {b }} 189$ | 92 | 47 | 90 | 96.7 | . 161 | 7570 | . 803 | 47 | 57 |
| ${ }^{\text {a }} 144$ | 95.4 | 590 | 588 | 587 | 583 | 565 | 499 | 383 | 162 | 92 | 40 | 38 | 33 | 52 | 389 | 90 | 39 | 109 | 96.5 | . 156 | 8200 | . 796 | 46 | 56 |
| 288 | 95.4 | 611 | 609 | 607 | 603 | 585 | 517 | 397 | 167 | 95 | 40 | 38 | 33 | 55 | 402 | 92 | 39 | 105 | 96.6 | . 155 | 8350 | . 793 | 46 | 56 |
| 375 | 95.1 | 627 | 625 | 623 | 619 | 600 | 530 | 408 | 172 | 96 | 40 | 38 | 44 | $97^{\prime}$ | 413 | 95 | 39 | 203 | 96.2 | . 153 | 8510 | . 787 | 45 | 55 |
| 127 | 95.1 | 656 | 654 | 652 | 648 | 631 | 568 | $\mathrm{b}_{245}$ | $\mathrm{b}_{81}$ | ${ }^{5} 5$ | 42 | 40 | 34 | 31 | $\mathrm{b}_{266}{ }^{\text {b }}$ | $\mathrm{b}_{54}$ | 38 | 94 | 96.5 | . 080 | 8080 | . 783 | 44 | 54 |
| 372 | 95.4 | 674 | 671 | 669 | 665 | 644 | 569 | 436 | 182 | 100 | 41 | 38 | 33 | 51 | 442 | ${ }^{99}$ | 39 | 108 | 96.7 | . 149 | 8810 | . 788 | 45 | 56 |
| (b) Stagnation temperature, $\mathrm{T}_{0}, 109.9 \mathrm{~K}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 1-280 | 109.1 | 155 | 154 | 155 | 154 | 153 | 146 | 135 | 109 | 88 | 67 | 60 | 51 | 50 | 135 | 89 | 67 | 57 | 95.9 | 0.568 | 2320 | 1.148 | 139 | 140 |
| 278 | 109.2 | 169 | 168 | 168 | 168 | 166 | 158 | 145 | 116 | 93 | 67 | 59 | 50 | 48 | 145 | 94 | 68 | 55 | 96.9 | . 551 | 2520 | 1.147 | 139 | 140 |
| $c_{52}$ | 110.3 | 173 | 172 | 173 | 172 | 170 | 163 | 149 | 119 | 96 | 71 | 63 | 52 | 43 | 149 | 97 | 71 | 49 | 93.2 | . 557 | 2490 | 1.172 | 148 | 149 |
| ${ }^{\text {d }} 42$ | 109.8 | 188 | 187 | 188 | 187 | 190 | 176 | 160 | 127 | 100 | 70 | 60 | 50 | 44 | 160 | 103 | 70 | 46 | 93.5 | . 533 | 2490 | 1.157 | 142 | 145 |
| ${ }^{\text {c }} 51$ | 109.2 | 203 | 202 | 203 | 202 | 199 | 189 | 170 | 134 | 106 | 71 | 65 | 70 | 75 | 170 | 107 | 71 | 83 | 101.1 | . 521 | 3020 | 1.140 | 136 | 140 |
| 276 | 108.9 | 254 | 253 | 253 | 252 | 248 | 231 | 202 | 145 | 119 | 82 | 66 | 55 | 63 | 203 | 120 | 82 | 75 | 98.1 | . 467 | 3790 | 1.122 | 129 | 137 |
| ${ }^{\text {c }} 48$ | 109.9 | 267 | 266 | 266 | 265 | 260 | 244 | 213 | 154 | 126 | 84 | 68 | 53 | 60 | 214 | 128 | 85 | 72 | 99.6 | . 474 | 3720 | 1.145 | 137 | 146 |
| $\mathrm{d}_{40}$ | 110.4 | 273 | 271 | 272 | 271 | 275 | 249 | 217 | 156 | 128 | $86^{\prime}$ | 69 | 52 | 50 | 219 | 132 | 86 | 64 | 98.6 | . 471 | 3550 | 1.153 | 141 | 150 |
|  | 109.1 | 313 | 312 | 312 | 310 | 304 | 280 | 239 | 157 | 126 | 90 | 73 | 60 | 72 | 241 | 127 | 90 | 86 | 101.7 | . 402 | 4550 | 1.118 | 128 | 139 |
| $\mathrm{d}_{38}$ | 109.9 | 345 | 343 | 343 | 342 | 345 | 308 | 260 | 167 | 133 | 94 | $76$ | $56$ | $55$ | 262 | 135 | 94 | 80 | 100.6 | . 385 | 4720 | 1.130 | 132 | 146 |
| $\mathrm{d}_{36}$ | 110.1 | 407 | 406 | 406 | 404 | 407 | 361 | 300 | 181 | 137 | 98 | $80$ | 58 | 58 | 302 | 139 | 98 | 87 | 102.3 | . 337 | 5310 | 1.123 | 130 | 147 |
| 272 | 109.8 | 447 | 446 | 445 | 443 | 432 | 393 | 323 | 188 | 137 | 97 | 82 | 62 | 81 | 327 | 139 | 98 | 109 | 104.9 | . 306 | 5900 | 1.112 | 126 | 145 |
| $\mathrm{d}_{34}$ | 109.8 | 517 | 516 | 515 | 513 | 516 | 452 | 366 | 201 | 141 | 96 | 84 | 60 | 70 | 372 | 143 | 96 | 104 | 104.2 | . 273 | 6330 | 1.100 | 122 | 145 |
| 269 | 110.0 | 567 | 565 | 563 | 561 | 546 | 493 | 398 | 214 | 144 | 96 | 87 | 89 | 127 | 405 | 147 | 98 | 206 | 109.9 | . 254 | 6920 | 1.097 | 121 | 147 |
| 267 | 110.0 | 619 | 617 | 615 | 612 | 695 | 535 | 429 | 230 | 148 | 93 | 87 | 75 | 114 | 437 | 150 | 96 | 190 | 109.8 | . 240 | 7410 | 1.089 | 118 | 146 |
| $\mathrm{d}_{44}$ | 110.9 | 663 | 661 | 660 | 657 | 660 | 575 | 460 | 241 | 159 | 96 | 90 | 63 | 63 | 468 | 159 | 99 | 114 | 105.5 | . 239 | 7160 | 1.103 | 123 | 155 |
| $\mathrm{c}_{54}$ | 111.3 | 665 | 663 | 662 | $\underbrace{658}$ | 640 | 576 | 462 | 243 | ! 161 | 98 | 90 | 63 | 67 | 469 | 159 | 100 | 119 | 106.6 | . 243 | 7320 | 1.111 | 126 | 158 |

[^0]TABLE VIII. - Continued.
(c) Stagnation temperature, $\mathrm{T}_{0}, 118.9 \mathrm{~K}$

| Reading | Stagna - | Stagna- | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\begin{gathered} \mathrm{P}_{\mathrm{b}},{ }_{2} / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure to stagnation pressure, $\mathrm{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}$ | Maximum mass flux, $G_{\text {max }}$, $\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} S_{0}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation <br> pressure <br> at stagna- <br> tion tem- <br> perature, $\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | tion temperature, ${ }^{T} \mathbf{0}$, K | tion <br> pressure, $\begin{aligned} & \mathbf{P}_{0^{\prime}} \\ & \mathrm{N} / \mathrm{cm}^{2} \end{aligned}$ | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ |  | $\mathrm{P}_{10}$ | $\mathrm{P}_{11}$ |  |  |  |  |  |  |  |  |  |  |  |
| 1-368 | 118.8 | 245 | 244 | 244 | 244 | 242 | 233 | 217 | 182 | 162 | 157 | 116 | 53 | 48 | 218 | 164 | 158 | 62 | 96.1 | 0.661 | 2770 | 1.381 | 234 | 236 |
| 249 | 118.2 | 246 | 245 | 246 | 245 | 242 | 232 | 214 | 173 | 145 | 139 | 127 | 59 | 40 | 214 | 146 | 139 | 59 | 96.6 | . 591 | 2710 | 1.363 | 226 | 229 |
| 360 | 119.6 | 247 | 247 | 247 | 247 | 245 | 239 | 228 | 203 | 185 | 127 | 81 | 55 | 71 | 228 | 187 | 128 | 78 | 100.1 | . 379 | 2060 | 1.408 | 245 | 246 |
| 354 | 119.2 | 247 | 247 | 247 | 247 | 245 | 236 | 222 | 189 | 171 | 163 | 105 | -97 | 102 | 223 | 174 | 164 | 108 | 105.2 | . 444 | 2420 | 1.394 | 239 | 241 |
| 154 | 118.9 | 270 | 270 | 270 | 269 | 266 | 254 | 234 | 189 | 160 | 152 | 132 | 63 | 77 | 235 | 160 | 152 | 93 | 102.9 | . 591 | 2920 | 1.375 | 231 | 238 |
| 245 | 118.5 | 314 | 314 | 313 | 312 | 307 | 291 | 263 | 204 | 156 | 123 | 121 | 78 | 50 | 264 | 159 | 124 | 75 | 99.9 | . 499 | 3560 | 1.344 | 218 | 233 |
| 156 | 119.2 | 356 | 355 | 355 | 354 | 348 | 328 | 293 | 224 | 175 | 127 | 124 | 82 | 55 | 295 | 178 | 128 | 92 | 102.6 | . 491 | 3900 | 1.347 | 219 | 241 |
| 218 | 119.0 | 390 | 389 | 389 | 388 | 381 | 357 | 315 | 234 | 188 | 114 | 111 | 87 | 60 | 318 | 191 | 115 | 94 | 103.3 | . 482 | 4290 | 1.332 | 213 | 239 |
| 217 | 119.0 | 423 | 422 | 422 | 420 | 413 | 384 | 336 | 241 | 195 | 120 | 103 | 90 | 65 | 339 | 197 | 120 | 110 | 105.6 | . 462 | 4660 | 1.321 | 208 | 239 |
| 252 | 118.9 | 441 | 440 | 439 | 437 | 428 | 399 | 345 | 243 | 195 | 121 | 93 | 88 | 67 | 349 | 196 | 122 | 95 | 102.7 | . 443 | 4900 | 1.314 | 205 | 237 |
| 254 | 118.4 | 460 | 459 | 458 | 456 | 446 | 414 | 356 | 243 | 194 | 123 | 95 | -76 | 69 | 360 | 195 | 124 | 95 | 102.9 | . 422 | 5160 | 1.297 | 198 | 231 |
| 159 | 118.6 | 461 | 460 | 459 | 457 | 448 | 415 | 357 | 245 | 196 | 123 | 95 | 5 82 | 69 | 361 | 197 | 124 | 102 | 104.0 | . 424 | 5120 | 1.303 | 200 | 234 |
| 258 | 118.4 | 482 | 481 | 480 | 478 | 467 | 433 | 371 | 249 | 197 | 129 | 100 | 72 | 68 | 375 | 198 | 130 | 95 | 103.3 | . 409 | 5310 | 1.291 | 196 | 231 |
| 215 | 119.0 | 517 | 516 | 515 | 513 | 502 | 462 | 393 | 259 | 201 | 131 | 101 | 77 | 85 | 397 | 202 | 132 | 124 | 107.5 | . 390 | 5560 | 1.297 | 198 | 238 |
| 162 | 118.9 | 568 | 567 | 565 | 563 | 550 | 505 | 423 | 265 | 202 | 134 | 103 | 73 | 100 | 429 | 202 | 134 | 145 | 110.0 | . 355 | 6100 | 1.283 | 192 | 237 |
| 164 | 119.2 | 614 | 612 | 610 | 607 | 594 | 543 | 453 | 278 | 206 | 137 | 106 | 76 |  | 459 | 207 | 137 | 158 | 111.5 | . 335 | 6450 | 1.281 | 191 | 241 |
| 212 | 118.8 | 678 | 676 | 674 | 671 | 655 | 596 | 492 | 289 | 205 | 134 | 107 | 74 | 80 | 497 | 205 | 135 | 125 | 107.5 | . 302 | 7030 | 1.260 | 183 | 237 |
| (d) Stagnation temperature, $\mathrm{T}_{0}, 124.3 \mathrm{~K}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 1-231 | 123.7 | 327 | 327 | 327 | 327 | 324 | 315 | 300 | 269 | 256 | 140 | 94 | 47 | 40 | 301 | 256 | 140 | 57 | 95.6 | 0.784 | 2390 | 1.519 | 289 | 302 |
| 353 | 124.0 | 329 | 328 | 328 | 328 | 326 | 317 | 303 | 274 | 261 | 132 | 89 | 45 | 41 | 304 | 264 | 133 | 55 | 95.2 | . 794 | 2300 | 1.528 | 292 | 305 |
| 288 | 124.8 | 330 | 330 | 330 | 329 | 327 | 319 | 306 | 253 | 209 | 125 | 85 | 42 | 38 | 307 | 209 | 126 | 53 | 94.6 | . 633 | 2151 | 1.576 | 308 | 317 |
| 82 | 123.7 | 346 | 346 | 345 | 345 | 342 | 330 | 310 | 270 | 250 | 165 | 112 | 27 | 43 | 311 | 251 | 165 | 64 | 97.2 | . 724 | 2770 | 1.498 | 282 | 301 |
| 232 | 124.7 | 358 | 358 | : 358 | 357 | 354 | 342 | 323 | 383 | 266 | 160 | 109 | 55 | 42 | 324 | 266 | 161 | 63 | 96.7 | . 744 | 2710 | 1.533 | 294 | 317 |
| 210 | 123.6 | 377 | 376 | 376 | 375 | 371 | 354 | 325 | 267 | 234 | 200 | 136 | 69 | 46 | 327 | 235 | 201 | 72 | 99.0 | . 621 | 3380 | 1.474 | 272 | 301 |
| 93 | 124.5 | 393 | 393 | 393 | 392 | 386 | 371 | 341 | 283 | 254 | 195 | 135 | - 69 | 47 | 343 | 254 | 195 | 73 | 99.5 | . 644 | 3340 | 1.495 | 281 | 314 |
| 236 | 124.7 | 410 | 410 | 409 | 409 | 403 | 383 | 349 | 280 | 242 | 207 | 145 | 75 | 50 | 351 | 243 | 208 | 79 | 100.4 | . 590 | 3650 | 1.489 | 278 | 316 |
| 207 | 124.2 | 426 | 425 | 425 | 424 | 418 | 396 | 357 | 278 | 234 | 212 | 152 | 82 | 100 | 358 | 236 | 213 | 131 | 108.1 | . 550 | 3970 | 1.464 | 268 | 309 |
| 96 | 123.6 | 456 | 455 | 454 | 453 | 446 | 419 | 372 | 280 | 227 | 196 | 157 | 89 | 68 | 375 | 230 | 196 | 123 | 107.3 | . 499 | 4380 | 1.432 | 255 | 300 |
| 200 | 123.7 | 486 | 484 | 483 | 482 | 474 | 443 | 390 | 285 | 231 | 189 | 160 | - 92 | 63 | 393 | 234 | 190 | 92 | 103.1 | . 476 | 4730 | 1.424 | 252 | 302 |
| 84 | 124.3 | 528 | 527 | 526 | 525 | 515 | 480 | 420 | 298 | 242 | 193 | 165 | 97 | 68 | 423 | 244 | 193 | 107 | 104.5 | . 458 | 5070 | 1.425 | 252 | 311 |
| 100 | 124.3 | 563 | 562 | 562 | 558 | 548 | 508 | 440 | 301 | 243 | 182 | 165 | 100 | 70 | 444 | 244 | 183 | 123 | 107.3 | . 431 | 5380 | 1.412 | 247 | 310 |
| 197 | 124.0 | 602 | 600 | 598 | 597 | 585 | 540 | 461 | 303 | 240 | 165 | 160 | 104 | 76 | 465 | 241 | 166 | 146 | 109.9 | . 399 | 5840 | 1.392 | 238 | 305 |
| 87 | 125.3 | 636 | 634 | 633 | 630 | 618 | 571 | 490 | 322 | 254 | 184 | 170 | - 105 | 75 | 494 | 254 | 185 | 118 | 106.3 | . 399 | 5880 | 1.415 | 248 | 325 |
| 105 | 125.3 | 636 | 635 | 633 | 630 | 618 | 571 | 490 | 323 | 255 | 187 | 171 | 105 | 74 | 495 | 256 | 188 | 114 | 106.0 | . 401 | 5850 | 1.417 | 249 | 326 |
| 103 | 124.5 | 640 | 638 | 636 | 633 | 621 | 572 | 488 | 316 | 245 | 163 | 159 | 105 | 75 | 492 | 246 | 163 | 118 | 106.4 | . 383 | 6040 | 1.394 | 239 | 313 |
| 101 | 124.6 | 670 | 669 | 667 | 663 | 650 | 599 | 509 | 323 | 250 | 164 | 162 | 107 | 76 | 513 | 250 | 164 | 121 | 106.9 | . 373 | 6230 | 1.390 | 237 | 315 |

TABLE VIII. - Concluded.
(e) Stagnation temperature, $\mathrm{T}_{0}, 129.5 \mathrm{~K}$

| Reading | Stagna- <br> tion temperature, $\mathrm{T}_{\mathrm{o}}$, K | Stagnation pressure, $\mathbf{P}_{\mathrm{c}}$, <br> $\mathrm{N} / \mathrm{cm}^{2}$ | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | $\begin{gathered} \text { Back- } \\ \text { pressure } \\ P_{b}{ }^{\prime} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure to stagnation pressure, $\mathrm{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}$ | Maximum mass flux, $G_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} \mathrm{S}_{\mathrm{o}}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\left(\begin{array}{l} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{array}\right.$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ | $\mathrm{P}_{10}{ }^{\text {P }}$ | $\mathrm{P}_{11} \mathrm{P}^{\text {P }}$ | $\mathrm{P}_{12}$ | $\mathrm{P}_{13}$ | $\mathrm{P}_{14}$ | ${ }^{P} 15$ |  |  |  |  |  |  |  |
| 1-191 | 126.4 | 339 | 339 | 339 | 338 | 337 | 327 | 310 | 256 | 204 | 112 | 73 | 73 | 95 | 311 | 204 | 113 | 105 | 104.8 | 0.600 | 1590 | ${ }^{\text {e }} 1.967$ | 330 | ( |
| 186 | 128.5 | 361 | 361 | 360 | 360 | 358 | 348 | 330 | 272 | 227 | 115 | 77 | 37 | 24 | 331 | 227 | 116 | 42 | 91.8 | . 628 | 1690 | ${ }^{\mathrm{e}} 1.992$ | 324 |  |
| 351 | 128.8 | 374 | 374 | 373 | 373 | 371 | 361 | 344 | 283 | 236 | 121 | 80 | 39 | 25 | 345 | 239 | 123 | 46 | 92.9 | . 631 | 1800 | ${ }^{\mathrm{e}_{1.922}}$ | 336 |  |
| 188 | 129.4 | 379 | 379 | 379 | 378 | 376 | 365 | 348 | 288 | 241 | 123 | 81 | 39 | 25 | 349 | 241 | 124 | 47 | 92.9 | . 635 | 1820 | ${ }^{e} 1.955$ | 333 |  |
| 172 | 129.6 | 402 | 402 | 401 | 401 | 398 | 387 | 369 | 312 | 254 | 141 | 95 | 48 | 32 | 370 | 254 | 142 | 55 | 94.9 | . 633 | 2190 | 1.798 | 344 |  |
| 347 | 129.3 | 406 | 405 | 405 | 405 | 402 | 390 | 371 | 317 | 261 | 148 | 99 | 50 | 37 | 372 | 264 | 149 | 59 | 95.4 | . 644 | 2310 | 1.745 | 339 |  |
| 174 | 129.8 | 423 | 422 | 422 | 422 | 418 | 406 | 385 | 340 | 280 | 157 | 106 | 53 | 35 | 386 | 281 | 159 | 60 | 96.1 | . 664 | 2480 | 1.731 | 338 |  |
| 329 | 129.3 | 433 | 433 | 432 | 432 | 427 | 414 | 390 | 341 | 324 | 169 | 115 | 58 | 38 | 392 | 328 | 171 | 61 | 96.7 | . 748 | 2760 | 1.668 | 330 |  |
| 331 | 129.4 | 444 | 444 | 444 | 443 | 438 | 423 | 397 | 342 | 322 | 180 | 123 | 62 | 41 | 398 | 326 | 181 | 64 | 97.2 | . 725 | 2950 | 1.654 | 328 |  |
| 183 | 130.1 | 462 | 461 | 461 | 460 | 455 | 439 | 410 | 350 | 326 | 188 | 128 | 66 | 81 | 412 | 327 | 189 | 106 | 104.7 | . 707 | 3050 | 1.661 | 329 |  |
| 344 | 129.0 | 475 | 474 | 473 | 473 | 467 | 446 | 411 | 338 | 307 | 206 | 144 | 75 | 113 | 413 | 311 | 208 | 137 | 108.9 | . 646 | 3530 | 1.598 | 315 |  |
| 176 | 129.8 | 512 | 511 | 510 | 509 | 502 | 476 | 435 | 346 | 310 | 223 | 157 | 82 | 55 | 437 | 311 | 223 | 86 | 101.4 | . 606 | 3900 | 1.597 | 315 |  |
| 343 | 129.5 | 545 | 545 | 543 | 542 | 534 | 505 | 453 | 346 | 300 | 234 | 167 | 91 | 60 | 456 | 305 | 235 | 88 | 102.0 | . 550 | 4380 | 1.562 | 304 |  |
| 97 | 130.1 | 564 | 563 | 561 | 560 | 552 | 520 | 466 | 351 | 304 | 239 | 171 | 93 | 62 | 468 | 305 | 239 | 95 | 103.0 | . 539 | 4490 | 1.568 | 306 |  |
| 340 | 129.6 | 597 | 596 | 594 | 592 | 583 | 547 | 485 | 355 | 298 | 246 | 178 | 137 | 209 | 488 | 303 | 246 | 238 | 119.2 | . 500 | 4910 | 1.537 | 296 |  |
| 332 | 129.9 | 638 | 638 | 636 | 633 | 621 | 582 | 511 | 362 | 298 | 246 | 183 | 184 | 232 | 515 | 303 | 247 | 275 | 122.0 | . 467 | 5250 | 1.526 | 292 |  |
| 314 | 129.5 | 663 | 662 | 660 | 657 | 646 | 601 | 523 | 360 | 292 | 242 | 182 | 106 | 74 | 527 | 296 | 244 | 117 | 105.1 | . 440 | 5580 | 1.506 | 285 |  |
| 178 | 129.9 | 680 | 679 | 677 | 675 | 662 | 616 | 536 | 368 | 296 | 244 | 184 | 108 | 74 | 540 | 296 | 245 | 113 | 105.5 | . 435 | 5650 | 1.510 | 286 |  |
| 336 | 284.7 | 343 | 343 | 343 | 342 | 339 | 327 | 305 | 240 | 194 | 86 | 52 | 76 | 85 | 306 | 196 | 82 | 107 | 280.4 | . 566 | 870 | 3.298 | (f) |  |
| 337 | 283.5 | 343 | 343 | 342 | 342 | 339 | 327 | 305 | 239 | 194 | 86 | 52 | 24 | 15 | 306 | 196 | 82 | 25 | 274.3 | . 565 | 874 | 3.293 | (f) | Y |

$\mathrm{e}_{\mathrm{S}_{0}}>\mathrm{S}_{\mathrm{c}}=1.813$.
$\mathrm{f}_{\text {Not applicable. }}$
(a) Stagnation temperature, $\mathrm{T}_{\mathrm{D}}, 89 \mathrm{~K}$

| Reading | Stagnation temperature, $\mathrm{T}_{0}$, K | Stagnation pressure,$\stackrel{\mathrm{P}_{\mathrm{o}}}{\mathrm{~N} / \mathrm{cm}_{2}}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Back pressure,$\begin{gathered} \mathrm{p}_{\mathrm{b}^{\prime}} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Tempera- <br> ture at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{o}$ | $\begin{aligned} & \text { Maximum } \\ & \text { mass flux, } \\ & \mathrm{G}_{\max } \\ & \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{aligned}$ | $\begin{aligned} & \text { Stagnation } \\ & \text { entropy, } \\ & \mathrm{S}_{0}, \\ & \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{aligned}$ | $\begin{gathered} \text { Saturation } \\ \text { pressure } \\ \text { at stagna- } \\ \text { tion en- } \\ \text { tropy, } \\ P_{\text {sat }}\left(S_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} P_{\mathrm{sat}}\left(\mathrm{~T}_{\mathrm{o}}\right) \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-457 | 87.5 | 562 | 559 | 534 | 173 | 57 | 25 | 21 | 34 | 191 | 198 | 199 | 88.4 | 0.045 | 8470 | 0.629 | 24 | 29 |
| 464 | 87.6 | 717 | 714 | 682 | 218 | 67 | 26 |  | 59 | 288 | 296 | 295 | 88.7 | . 036 | 9630 | . 619 | 23 |  |
| 907 | 87.7 | 773 | 769 | 734 | 233 |  | 27 |  | 44 | 275 | 285 | 285 | 89.1 | . 034 | 10060 | . 617 | 23 |  |
| 449 | 87.4 | 857 | 852 | 814 | 256 | 76 | 26 | $\dagger$ | 94 | 364 | 374 | 372 | 88.9 | . 030 | 10570 | . 605 | 22 | $\dagger$ |
| 459 | 90.3 | 353 | 353 | 337 | 122 | 50 | 31 | 24 | 32 | 104 | 113 | 115 | 90.7 | . 088 | 6490 | . 707 | 33 | 37 |
| 466 | 90.6 | 475 | 474 | 453 | 155 | 58 | 32 | 26 | 44 | 184 | 188 | 189 | 91.1 | . 067 | 7610 | . 703 | 33 | 38 |
| 452 | 90.5 | 632 | 630 | 602 | 199 | 67 | 32 | 25 | 41 | 216 | 224 | 224 | 91.4 | . 050 | 8890 | . 689 | 31 | 38 |

(b) Stagnation temperature, $\mathrm{T}_{0}, 97 \mathrm{~K}$

| 117-485 | 94.3 | 444 | 443 | 424 | 156 | 67 | 42 | 33 | 68 | 197 | 201 | 202 | 94.6 | 0.096 | 7160 | 0.785 | 44 | 51 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 462 | 96.8 | 156 | 157 | 152 | 83 | 61 | 53 | 49 | 61 | 94 | 93 | 96 | 96.6 | . 337 | 3540 | . 868 | 60 | 62 |
| 841 | 96.5 | 183 | 185 | 178 | 91 | 61 | 52 | 45 | 53 | 92 | 93 | 96 | 96.3 | . 283 | 4050 | . 858 | 58 | 61 |
| 876 | 96.3 | 289 | 289 | 277 | 120 | 66 | 49 | 34 | 37 | 51 | 73 | 78 | 96.2 | . 171 | 5500 | . 843 | 55 | 60 |
| 735 | 96.4 | 433 | 432 | 414 | 157 | 74 | 48 | 37 | 82 | 205 | 209 | 211 | 96.6 | . 112 | 6950 | . 831 | 52 | 60 |
| 490 | 96.4 | 509 | 506 | 485 | 180 | 78 | 49 | 38 | 125 | 256 | 261 | 261 | 96.8 | . 097 | 7620 | . 823 | 51 | 60 |
| 741 | 96.5 | 681 | 678 | 649 | 225 | 87 | 48 | 37 | 118 | 317 | 324 | 323 | 97.0 | . 070 | 8940 | . 810 | 49 | 61 |
| 842 | 98.3 | 148 | 150 | 145 | 87 | 66 | 58 | 55 | 63 | 91 | 91 | 95 | 97.9 | . 392 | 3280 | . 902 | 67 | 69 |
| 877 | 98.6 | 216 | 217 | 209 | 106 | 73 | 60 | 39 | 48 | 64 | 77 | 81 | 98.4 | . 277 | 4400 | . 901 | 67 | 71 |
| 847 | 98.5 | 218 | 219 | 211 | 107 | 71 | 60 | 37 | 44 | 55 | 68 | 73 | 98.2 | . 274 | 4410 | . 898 | 66 | 70 |
| 488 | 99.2 | 223 | 224 | 216 | [111 | 74 | 63 | 49 | 64 | 112 | 113 | 116 | 99.1 | . 281 | 4420 | . 913 | 69 | 74 |
| 736 | 98.6 | 255 | 255 | 246 | 117 | 73 | 59 | 44 | 64 | 122 | 124 | 127 | 98.5 | . 230 | 4900 | . 896 | 66 | 71 |
| 742 | 98.1 | 492 | 490 | 470 | 178 | 81 | 54 | 40 | 91 | 231 | 236 | 237 | 98.3 | . 110 | 7370 | . 861 | 58 | 69 |
| 783 | 99.8 | 778 | 773 | 738 | 262 | 101 | 57 | 41 | 79 | 303 | 310 | 308 | 100.5 | . 073 | 9430 | . 868 | 60 | 77 |

(c) Stagnation temperature, $\mathrm{T}_{0}, 104 \mathrm{~K}$

| 117-786 | 103.2 | 361 | 359 | 344 | 147 | 95 | 75 | 45 | 69 | 137 | 142 | 144 | 103.1 | 0.209 | 5750 | 0.982 | 86 | 97 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 797 | 103.2 | 375 | 373 | 357 | 161 | 97 | 75 | 45 | 72 | 150 | 153 | 154 | 103.1 | 200 | 5910 | . 980 | 86 | 97 |
| 790 | 103.2 | 613 | 609 | 582 | 226 | 105 | 70 | 47 | 90 | 260 | 265 | 264 | 103.3 | . 115 | 8020 | . 953 | 79 | 97 |
| 801 | 103.2 | 621 | 616 | 589 | 228 | 106 | 71 | 47 | 87 | 254 | 260 | 258 | 103.3 | . 114 | 8070 | . 952 | 79 | 97 |
| 802 | 103.2 | 784 | 778 | 744 | 275 | 114 | 69 | 47 | 94 | 318 | 326 | 322 | 103.5 | . 088 | 9250 | . 935 | 75 | 97 |
| 500 | 106.5 | 357 | 357 | 345 | 173 | 114 | 98 | 63 | 103 | 181 | 184 | 186 | 106.1 | . 270 | 5480 | 1.053 | 107 | 119 |
| 502 | 106.4 | 492 | 491 | 472 | 208 | 117 | 91 | 63 | 139 | 258 | 262 | 263 | 106.1 | . 185 | 6810 | 1.033 | 101 | 118 |
| 699 | 106.6 | 837 | 831 | 798 | 306 | 138 | 90 | 58 | 101 | 316 | 327 | 325 | 107.0 | . 108 | 9330 | . 997 | 90 | 120 |

TABLE IX. - Continued.
(d) Stagnation temperature, $T_{0}, 111 \mathrm{~K}$

| Reading | Stagnation temperature, To, K | Stagnation pressure, <br>  | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Backpressure,$\underset{\mathrm{N} / \mathrm{cm}^{2}}{\mathrm{P}_{\mathrm{t}}}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max }, \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | $\begin{gathered} \text { Stagnation } \\ \text { entropy }, \\ S_{0}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagna tion entropy, $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{S}_{0}\right)$,$\mathrm{N} / \mathrm{cm}^{2}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}\left(\mathbf{T}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-637 | 111.9 | 422 | 421 | 407 | 216 | 151 | 129 | 57 | 85 | 128 | 149 | 151 | 109.7 | 0.306 | 5590 | 1.160 | 143 | 164 |
| 710 | 111.8 | 427 | 426 | 413 | 217 | 151 | 129 | 59 | 91 | 141 | 160 | 163 | 111.0 | . 303 | 5660 | 1.157 | 142 | 163 |
| 711 | 111.8 | 492 | 491 | 474 | 235 | 154 | 128 | 61 | 83 | 135 | 163 | 166 | 111.0 | . 260 | 6290 | 1.146 | 138 | 163 |
| 51.2 | 110.7 | 560 | 559 | 539 | 249 | 149 | 119 | 63 | 114 | 236 | 244 | 244 | 110.3 | . 213 | 6980 | 1.113 | 126 | 153 |
| 712 | 111.9 | 570 | 567 | 548 | 256 | 156 | 125 | 63 | 76 | 129 | 166 | 169 | 111.0 | . 219 | 6950 | 1.136 | 134 | 164 |
| 638 | 111.0 | 608 | 604 | 582 | 261 | 151 | 117 | 62 | 65 | 111 | 153 | 158 | 110.2 | . 193 | 7320 | 1.112 | 126 | 155 |
| 639 | 110.6 | 827 | 822 | 790 | 318 | 155 | 109 | 63 | 57 | 100 | 151 | 164 | 110.0 | . 131 | 8950 | 1.076 | 114 | 152 |

(e) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 117 \mathrm{~K}$

| 117-827 | 116.3 | 215 | 218 | 213 | 148 | 117 | 109 | 105 | 75 | 111 | 115 | 119 | 105.1 | 0.507 | 3040 | 1.318 | 207 | 208 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 642 | 116.6 | 348 | 348 | 338 | 196 | 125 | 110 | 110 | 146 | 200 | 204 | 206 | 115.6 | . 315 | 4610 | 1.284 | 192 | 211 |
| 643 | 116.7 | 496 | 494 | 479 | 262 | 182 | 157 | 74 | 123 | 199 | 211 | 213 | 115.3 | . 316 | 5820 | 1.251 | 178 | 212 |
| 644 | 116.7 | 641 | 638 | 616 | 298 | 187 | 154 | 70 | 113 | 217 | 238 | 239 | 115.2 | . 240 | 7080 | 1.223 | 167 | 212 |
| 645 | 116.9 | 777 | 774 | 747 | 335 | 190 | 147 | 74 | 118 | 259 | 281 | 280 | 115.4 | . 189 | 8080 | 1.205 | 160 | 215 |

(f) Stagnation temperature, $T_{0}, 120 \mathrm{~K}$

| 117-526 | 119.9 | 277 | 278 | 273 | 205 | 183 | 177 | 159 | 153 | 181 | 183 | 185 | 113.3 | 0.641 | 3020 | 1.402 | 243 | 250 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 854 | 119.8 | 296 | 296 | 290 | 206 | 174 | 166 | 156 | 110 | 156 | 159 | 162 | 111.2 | . 560 | 3420 | 1.389 | 238 | 249 |
| 853 | 119.7 | 350 | 349 | 340 | 212 | 154 | 142 | 136 | 125 | 183 | 187 | 190 | 114.2 | . 407 | 4290 | 1.364 | 226 | 248 |
| 527 | 119.3 | 418 | 418 | 406 | 241 | 164 | 139 | 108 | 154 | 218 | 224 | 226 | 117.6 | . 332 | 4950 | 1.331 | 212 | 243 |
| 852 | 119.9 | 447 | 446 | 433 | 258 | 177 | 149 | 113 | 117 | 200 | 207 | 209 | 116.1 | . 333 | 5120 | 1.337 | 215 | 250 |
| 851 | 119.6 | 546 | 542 | 526 | 290 | 203 | 173 | 81 | 108 | 200 | 214 | 216 | 116.7 | . 318 | 6010 | 1.304 | 201 | 246 |
| 528 | 119.5 | 571 | 568 | 551 | 295 | 205 | 176 | 72 | 125 | 210 | 229 | 230 | 117.3 | . 309 | 6240 | 1.296 | 197 | 245 |
| 647 | 119.3 | 604 | 601 | 582 | 302 | 204 | 174 | 71 | 134 | 226 | 244 | 244 | 117.2 | . 287 | 6480 | 1.285 | 192 | 243 |
| 529 | 119.7 | 708 | 706 | 682 | 333 | 209 | 173 | 77 | 113 | 210 | 244 | 244 | 117.3 | . 244 | 7340 | 1.273 | 187 | 248 |
| 530 | 120.6 | 818 | 814 | , 786 | 369 | , 216 | 172 | 85 | 197 | 388 | 397 | 395 | 119.1 | . 211 | 8040 | 1.271 | 187 | 259 |

(g) Stagnation temperature, $\mathrm{T}_{0}, 122 \mathrm{~K}$

| 117-863 | 121.4 | 302 | 302 | 297 | 236 | 212 | 205 | 148 | 56 | 108 | 116 | 119 | 105.5 | 0.677 | 2910 | 1.441 | 259 | 270 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 538 | 121.8 | 346 | 346 | 339 | 245 | 206 | 196 | 169 | 211 | 243 | 246 | 247 | 118.9 | . 568 | 3590 | 1.429 | 254 | 275 |
| 862 | 121.3 | 387 | 386 | 376 | 243 | 183 | 166 | 155 | 79 | 151 | 162 | 164 | 111.3 | . 429 | 4360 | 1.394 | 240 | 268 |
| 534 | 121.2 | 489 | 488 | 475 | 280 | 198 | 168 | 127 | 217 | 290 | 294 | 295 | 119.6 | . 344 | 5320 | 1.357 | 223 | 267 |
| 540 | 122.3 | 501 | 500 | 486 | 289 | 205 | 176 | 143 | 198 | 275 | 281 | 281 | 120.2 | . 352 | 5320 | 1.381 | 234 | 282 |
| 442 | 122.3 | 708 | 703 | 680 | 347 | 226 | 189 | 112 | 290 | 411 | 417 | 413 | 120.8 | . 267 | 7020 | 1.328 | 211 | 282 |

TABLE LX. - Continued.
(h) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 124 \mathrm{~K}$

| Reading | Stagnation temperature, To, K | Stagnation pressure,$\begin{gathered} P_{0}, \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Back pressure,$\begin{gathered} \mathrm{P}_{\mathrm{b}},{ }_{\mathrm{N}} / \mathrm{cm} 2 \end{gathered}$ | Tempera- <br> ture at <br> back condition, $T_{b}$, K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} G_{\max } \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | Stagnation entropy,$\begin{gathered} \mathrm{S}_{\mathrm{O}} \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy, $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{S}_{\mathrm{o}}\right)$,$\mathrm{N} / \mathrm{cm}^{2}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $P_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-550 | 124.3 | 359 | 359 | 354 | 284 | 259 | 251 | 140 | 96 | 124 | 136 | 139 | 108.2 | 0.699 | 2980 | 1.514 | 288 | 311 |
| 548 | 124.4 | 506 | 504 | 491 | 306 | 226 | 200 | 174 | 161 | 245 | 252 | 254 | 120.1 | . 394 | 5060 | 1.435 | 257 | 312 |
| 547 | 124.4 | 558 | 556 | 541 | 321 | 232 | 201 | 165 | 152 | 248 | 258 | 259 | 120.5 | . 361 | 5530 | 1.417 | 249 | 312 |
| 546 | 124.5 | 630 | 627 | 609 | 341 | 241 | 209 | 153 | 181 | 285 | 295 | 295 | 121.4 | . 332 | 6170 | 1.398 | 241 | 314 |
| 545 | 124.4 | 701 | 698 | 677 | 361 | 242 | 207 | 198 | 415 | 485 | 492 | 490 | 123.1 | . 296 | 6760 | 1.377 | 232 | 312 |
| 544 | 124.5 | 809 | 805 | 779 | 389 | 243 | 201 | 181 | 435 | 528 | 536 | 534 | 123.2 | . 249 | 7580 | 1.355 | 223 | 314 |

(i) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 125 \mathrm{~K}$

| 117-569 | 125.1 | 356 | 356 | 352 | 298 | 279 | 273 | 119 | 137 | 179 | 182 | 185 | 113.8 | 0.765 | 2560 | 1.556 | 302 | 323 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 568 | 124.9 | 429 | 428 | 419 | 296 | 247 | 233 | 175 | 235 | 284 | 286 | 289 | 122.6 | . 542 | 4000 | 1.487 | 278 | 320 |
| 553 | 125.0 | 434 | 434 | 424 | 298 | 246 | 232 | 178 | 187 | 249 | 253 | 253 | 120.0 | . 535 | 4080 | 1.487 | 278 | 321 |
| 567 | 125.1 | 496 | 495 | 482 | 309 | 236 | 212 | 181 | 250 | 317 | 320 | 322 | 122.8 | . 428 | 4830 | 1.460 | 267 | 323 |
| 566 | 125.3 | 634 | 632 | 614 | 348 | 247 | 212 | 165 | 269 | 375 | 379 | 380 | 123.1 | . 335 | 6100 | 1.417 | 249 | 326 |
| 565 | 125.3 | 771 | 768 | 743 | 383 | 249 | 208 | 141 | 280 | 427 | 433 | 433 | 123.1 | . 270 | 7160 | 1.381 | 234 | 326 |
| 564 | 125.1 | 890 | 887 | 856 | 413 | 248 | 197 | 120 | 364 | 510 | 518 | 517 | 123.3 | . 221 | 8060 | 1.352 | 221 | 323 |

(j) Stagnation temperature, $T_{0}, 126 \mathrm{~K}$

| 117-600 | 126.1 | 387 | 388 | 382 | 312 | 287 | 279 | 135 | 55 | 117 | 129 | 132 | 107.1 | 0.721 | 2910 | 1.567 | 306 | 338 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 582 | 125.8 | 428 | 428 | 419 | 308 | 264 | 251 | 169 | 122 | 178 | 187 | 190 | 114.1 | . 587 | 3770 | 1.519 | 290 | 334 |
| 599 | 125.8 | 432 | 432 | 423 | 309 | 264 | 251 | 170 | 117 | 146 | 161 | 165 | 111.4 | . 581 | 3810 | 1.517 | 289 | 334 |
| 581 | 126.0 | 500 | 499 | 487 | 320 | 251 | 230 | 187 | 260 | 325 | 327 | 329 | 123.6 | . 459 | 4690 | 1.484 | 277 | 337 |
| 598 | 126.0 | 565 | 564 | 549 | 335 | 249 | 217 | 184 | 90 | 191 | 213 | 214 | 116.5 | . 385 | 5370 | 1.457 | 266 | 337 |
| 437 | 126.1 | 572 | 569 | 553 | 334 | 248 | 217 | 362 | 451 | 488 | 493 | 490 | 125.4 | . 379 | 5450 | 1.456 | 266 | 338 |
| 605 | 125.8 | 573 | 572 | 556 | 332 | 247 | 214 | 181 | 243 | 333 | 337 | 339 | 123.1 | . 374 | 5490 | 1.449 | 263 | 334 |
| 436 | 125.9 | 632 | 629 | 611 | 350 | 250 | 216 | 364 | 477 | 523 | 528 | 252 | 125.2 | . 342 | 6010 | 1.432 | 256 | 335 |
| 597 | 126.2 | 699 | 697 | 676 | 370 | 256 | 219 | 167 | 98 | 212 | 245 | 245 | 119.3 | . 313 | 5610 | 1.419 | 250 | 340 |
| 604 | 126.0 | 700 | 699 | 677 | 368 | 255 | 217 | 164 | 236 | 358 | 365 | 365 | 123.0 | . 311 | 6540 | 1.414 | 248 | 337 |
| 603 | 125.8 | 864 | 861 | 832 | 409 | 253 | 205 | 134 | 227 | 395 | 405 | 404 | 122.8 | . 237 | 7810 | 1.372 | 230 | 337 |
| 577 | 126.2 | 873 | 870 | 840 | 417 | 256 | 207 | 178 | 467 | 566 | 574 | 572 | 124.6 | . 237 | 7840 | 1.378 | 233 | 340 |

TABLE IX. - Continued.
(k) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 127 \mathrm{~K}$

| Reading | Stagnationtempera-ture,$\mathrm{T}_{0}$,K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{\mathrm{o}},{ }_{2} \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | $\begin{gathered} \text { Back- } \\ \text { pressure, } \\ P_{b}, \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Tempera- <br> ture at <br> back con- <br> dition, $T_{b},$ K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max } \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | $\begin{gathered} \text { Stagnation } \\ \text { entropy, } \\ \mathrm{S}_{\mathrm{o}}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathbf{P}_{\mathrm{sat}\left(\mathrm{~S}_{\mathrm{o}}\right)} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-575 | 126.9 | 392 | 391 | 386 | 324 | 303 | 295 | 124 | 87 | 111 | 123 | 127 | 106.5 | 0.753 | 2680 | 1.602 | 316 | (a) |
| 612 | 127.4 | 395 | 395 | 390 | 331 | 312 | 300 | 119 | 81 | 106 | 118 | 122 | 105.7 | . 760 | 2570 | 1.626 | 322 |  |
| 611 | 127.2 | 426 | 426 | 418 | 327 | 294 | 283 | 153 | 103 | 131 | 145 | 149 | 109.4 | . 665 | 3310 | 1.574 | 308 |  |
| 574 | 126.9 | 503 | 502 | 490 | 329 | 265 | 245 | 186 | 142 | 182 | 200 | 203 | 115.4 | . 487 | 4540 | 1. 509 | 286 |  |
| 610 | 127.1 | 562 | 562 | 547 | 343 | 262 | 233 | 192 | 148 | 188 | 211 | 214 | 116.6 | . 415 | 5170 | 1.486 | 278 |  |
| 573 | 127.0 | 639 | 637 | 619 | 361 | 262 | 227 | 186 | 163 | 217 | 244 | 246 | 119.5 | . 355 | 5900 | 1.455 | 265 |  |
| 609 | 127.4 | 699 | 698 | 677 | 377 | 266 | 228 | 182 | 164 | 252 | 277 | 278 | 122.0 | . 326 | 6370 | 1.445 | 261 |  |
| 572 | 127.0 | 773 | 770 | 746 | 394 | 262 | 220 | 166 | 105 | 250 ! | ! 281 | 281 | 122.1 | . 285 | 6980 | 1.417 | 249 |  |
| 608 | 127.4 | 845 | 841 | 814 | 415 | 265 | 218 | 160 | 140 | 295 | 322 | 321 | 122.7 | . 258 | 7490 | 1.408 | 246 |  |
| 571 | 126.8 | 897 | 892 | 862 | 426 | 260 | 209 | 143 | 106 | 247 | 293 | 292 | 121.8 | . 233 | 7910 | 1. 385 | 236 | $\dagger$ |

(l) Stagnation temperature, $T_{0}, 130 \mathrm{~K}$

| 117-894 | 130.5 | 418 | 419 | 414 | 354 | 281 | 253 | 111 | 48 | 104 | 117 | 120 | 105.5 | 0.604 | 2320 | 1.809 | 341 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 899 | 130.7 | 428 | 427 | 422 | 359 | 293 | 263 | 117 | 48 | 107 | 120 | 123 | 105.8 | . 613 | 2420 | 1.784 | 341 |  |
| 624 | 130.1 | 444 | 444 | 437 | 361 | 334 | 322 | 135 | 90 | 119 | 135 | 138 | 108.0 | . 725 | 2800 | 1.695 | 334 |  |
| 687 | 130.7 | 444 | 444 | 439 | 366 | 338 | 307 | 129 | 53 | 118 | 132 | 135 | 107.6 | . 692 | 2690 | 1.733 | 338 |  |
| 757 | 130.3 | 453 | 453 | 446 | 364 | 333 | 321 | 141 | 132 | 198 | 203 | 206 | 115.8 | . 710 | 3000 | 1.689 | 333 |  |
| 680 | 130.1 | 463 | 463 | 457 | 363 | 328 | 316 | 151 | 102 | 134 | 151 | 155 | 110.1 | . 681 | 3200 | 1.661 | 329 |  |
| 679 | 130.6 | 520 | 502 | 493 | 370 | 324 | 308 | 171 | 118 | 154 | 173 | 177 | 112.7 | . 613 | 3730 | 1.635 | 323 |  |
| 770 | 130.3 | 527 | 526 | 516 | 369 | 312 | 294 | 181 | 76 | 170 | 187 | 189 | 114.0 | . 558 | 4210 | 1.601 | 315 |  |
| 884 | 130.3 | 561 | 560 | 548 | 375 | 306 | 284 | 191 | 206 | 293 | 298 | 300 | 123.6 | . 507 | 4590 | 1.577 | 309 |  |
| 622 | 130.4 | 568 | 566 | 554 | 375 | 306 | 283 | 192 | 141 | 182 | 204 | 207 | 115.9 | . 499 | 4630 | 1.576 | 308 | $\geqslant$ |

(m) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 131 \mathrm{~K}$

| 117-945 | 130.9 | 385 | 384 | 380 | 318 | 256 | 230 | 94 | 107 | 158 | 161 | 164 | 110.0 | 0.598 | 1730 | ${ }^{\mathrm{b}} 2.026$ | 317 | a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 932 | 130.9 | 439 | 438 | 433 | 364 | 305 | 280 | 123 | 83 | 146 | 155 | 157 | 110.5 | . 637 | 2560 | 1.762 | 340 |  |
| 811 | 130.8 | 449 | 447 | 440 | 366 | 338 | 313 | 128 | 78 | 143 | 152 | 153 | 110.3 | . 699 | 2660 | 1.727 | 338 |  |
| 760 | 131.3 | 462 | 462 | 456 | 372 | 340 | 327 | 140 | 208 | 259 | 261 | 264 | 120.9 | . 709 | 2930 | 1.728 | 338 |  |
| 883 | 131.2 | 700 | 698 | 679 | 411 | 303 | 267 | 211 | 238 | 354 | 361 | 362 | 126.0 | . 381 | 5850 | 1.533 | 295 |  |
| 621 | 130.9 | 708 | 706 | 687 | 407 | 298 | 261 | 210 | 98 | 223 | 252 | 253 | 120.0 | . 368 | 5960 | 1.523 | 291 |  |
| 620 | 131.1 | 863 | 861 | 834 | 448 | 295 | 246 | 201 | 106 | 246 | 289 | 289 | 122.8 | . 285 | 7180 | 1.478 | 275 | $\psi$ |

${ }^{\text {Not applicable. }}$
$\mathrm{b}_{\mathrm{S}_{0}}>\mathrm{S}_{\mathrm{c}}=1.813$.

| Reading | Stagnation temperature, To, K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{\mathrm{o}}{ }_{2} \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Backpressure, $P_{b}$, $\mathrm{N} / \mathrm{cm}^{2}$ | Temperature at back condition, $T_{b}$, K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max } \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \max \end{gathered}$ | Stagnation entropy,$\begin{gathered} S_{0} \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation Saturation pressure pressure at stagna- at stagnation en- tion temtropy, perature, $P_{\text {sat }}\left(\mathrm{S}_{0}\right), \quad \mathrm{P}_{\text {sat }}\left(\mathrm{T}_{0}\right)$, $\mathrm{N} / \mathrm{cm}^{2} \mid \mathrm{N} / \mathrm{cm}^{2}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-900 | 132.2 | 468 | 466 | 460 | 378 | 344 | 329 | 136 | 52 | 123 | 139 | 140 | 108.3 | 0.704 | 2800 | 1.767 | 340 | (a) |
| 928 | 131.8 | 481 | 482 | 474 | 378 | 340 | 327 | 151 | 135 | 208 | 214 | 216 | 116.7 | . 678 | 3110 | 1.718 | 337 |  |
| 929 | 132.2 | 500 | 499 | ! 491 | \| 383 | 340 | 325 | 161 | 143 | 219 | 225 | 227 | 117.8 | . 650 | 3340 | 1.706 | 335 |  |
| 809 | 132.1 | 559 | 556 | 544 | 389 | 326 | 306 | 184 | 122 | 211 | 222 | 222 | 117.5 | . 547 | 4150 | 1.636 | 324 |  |
| 684 | 131.9 | 570 | 569 | 558 | 391 | 325 | 303 | 190 | 80 | 180 | 200 | 201 | 115.3 | . 531 | 4380 | 1.621 | 320 |  |
| 882 | 132.0 | 901 | 897 | 870 | 469 | 304 | 251 | 206 | 236 | 395 | 407 | 406 | 126.6 | . 278 | 7350 | 1.487 | 278 |  |
| 768 | 132.1 | 918 | 914 | 887 | 473 | 303 | 250 | 205 | 108 | 255 | 301 | 301 | 123.7 | . 272 | 7560 | 1.484 | 277 | 1 |

(0) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 133 \mathrm{~K}$

| 117-893 | 132.4 | 476 | 476 | 470 | 381 | 345 | 331 | 143 | 92 | 127 | 145 | 148 | 109.3 | 0.695 | 2920 | 1.760 | 340 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 915 | 132.9 | 476 | 476 | 469 | 382 | 346 | 331 | 139 | 94 | 166 | 176 | 178 | 112.9 | . 694 | 2910 | 1.789 | 341 |  |
| 688 | 132.8 | 565 | 563 | 553 | 398 | 335 | 314 | 187 | 76 | 174 | 194 | 195 | 114.6 | . 555 | 4150 | 1.655 | 328 |  |
| 685 | 133.0 | 669 | 665 | 650 | 419 | 324 | 293 | 209 | 92 | 209 | 235 | 235 | 118.5 | . 439 | 5250 | 1.591 | 313 |  |
| 808 | 133.1 | 682 | 678 | 660 | 421 | 321 | 289 | 209 | 138 | 259 | 273 | 272 | 121.7 | . 423 | 5330 | 1.588 | 312 | $\dagger$ |

(p) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 134 \mathrm{~K}$

| 117-662 | 133.7 | 483 | 482 | 476 | 387 | 348 | 332 | 139 | 53 | 129 | 144 | 147 | 109.2 | 0.687 | 2750 | ${ }^{\mathrm{b}} 1.816$ | 341 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 661 | 133.8 | 486 | 486 | 479 | 388 | 349 | 332 | 140 | 65 | 142 | 155 | 158 | 110.6 | . 684 | 2780 | ${ }^{\text {b }} 1.814$ | 341 |  |
| 663 | 134.1 | 490 | 489 | 483 | 390 | 348 | 333 | 141 | 53 | 132 | 148 | 150 | 109.6 | . 679 | 2790 | $\mathrm{b}_{1.821}$ | 341 |  |
| 892 | 134.2 | 559 | 558 | 549 | 407 | 347 | 326 | 181 | 121 | 163 | 185 | 188 | 113.9 | . 583 | 3810 | 1.711 | 336 |  |
| 759 | 133.9 | 667 | 666 | 651 | 426 | 332 | 302 | 208 | 307 | 403 | 407 | 409 | 128.6 | . 452 | 5180 | 1.616 | 319 |  |
| 754 | 133.8 | 845 | 840 | 417 | 468 | 320 | 273 | 221 | 238 | 383 | 393 | 394 | 127.3 | . 324 | 6780 | 1.540 | 297 |  |
| 807 | 133.7 | 846 | 838 | 314 | 467 | 318 | 271 | 219 | 136 | 295 | 320 | 317 | 125.0 | . 321 | 6650 | 1.537 | 296 | $\dagger$ |

(q) Stagnation temperature, $\mathrm{T}_{0}, 135 \mathrm{~K}$

| 117-946 | 135.9 | 429 | 429 | 423 | 339 | 290 | 275 | 110 | 125 | 186 | 189 | 192 | 114.4 | 0.644 | 1960 | $\mathrm{b}_{2.088}$ | 298 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 817 | 134.9 | 431 | 429 | 422 | 343 | 302 | 279 | 108 | 93 | 154 | 159 | 161 | 111.2 | . 647 | 1910 | $\mathrm{b}_{2.043}$ | 312 |  |
| 942 | 135.2 | 460 | 460 | 454 | 370 | 330 | 309 | 123 | 106 | 170 | 177 | 180 | 113.1 | . 672 | 2250 | ${ }^{\mathrm{b}} 1.964$ | 330 |  |
| 664 | 135.1 | 521 | 519 | 512 | 401 | 351 | 332 | 157 | 58 | 144 | 162 | 164 | 111.3 | . 638 | 3090 | 1.805 | 341 |  |
| 902 | 135.2 | 577 | 574 | 564 | 417 | 350 | 328 | 184 | 72 | 169 | 190 | 190 | 114.2 | . 568 | 3880 | 1.727 | 338 |  |
| 922 | 136.1 | 577 | 574 | 564 | 420 | 353 | 330 | 180 | 119 | 216 | 228 | 228 | 117.9 | . 572 | 3710 | 1.760 | 340 |  |
| 758 | 134.9 | 848 | 842 | 420 | 477 | 329 | 283 | 226 | 310 | 459 | 466 | 466 | 129.5 | . 334 | 6640 | 1.562 | 304 |  |
| 689 | 136.0 | 860 | 855 | 833 | 490 | 330 | 292 | 230 | 105 | 250 | 288 | 287 | 122.7 | . 340 | 6470 | 1. 581 | 310 | 1 |

[^1]$\mathrm{b}_{\mathrm{o}}>\mathrm{S}_{\mathrm{c}}=1.813$.

TABLE IX. - Concluded.
(r) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 138 \mathrm{~K}$

| Reading | Stagnation temperature, To, K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{\mathrm{o}}, \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | $\begin{gathered} \text { Back- } \\ \text { pressure } \\ P_{b}, \\ N / \mathrm{cm}^{2} \end{gathered}$ | Tempera- <br> ture at <br> back con- <br> dition, $\begin{gathered} T_{b}, \\ { }_{K} \end{gathered}$ | Ratio of throat pressure at stagnation pressure, $\mathrm{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\text {max }} \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ | $\begin{aligned} & \text { Stagnation } \\ & \text { entropy, } \\ & \mathrm{S}_{0}, \\ & \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{aligned}$ | Saturation pressure at stagnation entropy, $\mathrm{P}_{\text {sat }}\left(\mathrm{S}_{\mathrm{o}}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ | Saturation <br> pressure <br> at stagna- <br> tion tem- <br> perature, $\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-940 | 138.0 | 559 | 557 | 549 | 420 | 355 | 331 | 172 | 210 | 287 | 291 | 293 | 123.1 | 0.592 | 3180 | $\mathrm{b}_{1.858}$ | 341 | (a) |
| 658 | 138.2 | 692 | 690 | 677 | 463 | 363 | 327 | 207 | 269 | 373 | 378 | 379 | 128.5 | . 473 | 4630 | 1.718 | 337 |  |
| 912 | 138.2 | 765 | 763 | 745 | 482 | 361 | 319 | 223 | 162 | 297 | 313 | 314 | 124.6 | . 416 | 5370 | 1.672 | 331 |  |
| 890 | 138.2 | 928 | 925 | 900 | 528 | 358 | 301 | 237 | 179 | 258 | 303 | 304 | 123.9 | . 325 | 6700 | 1.604 | 316 | $\gamma$ |

(s) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 140 \mathrm{~K}$

| 117-669 | 139.8 | 536 | 535 | 528 | 408 | 339 | 317 | 154 | 55 | 138 | 159 | 161 | 111.0 | 0.591 | 2670 | $\mathrm{b}_{1.968}$ | 330 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 668 | 139.3 | 596 | 595 | 586 | 439 | 361 | 332 | 182 | 67 | 167 | 189 | 190 | 114.2 | . 558 | 3400 | $\mathrm{b}_{1.849}$ | 341 |  |
| 657 | 139.1 | 766 | 763 | 747 | 490 | 368 | 324 | 223 | 283 | 406 | 411 | 412 | 130.0 | . 423 | 5200 | 1.694 | 334 |  |
| 923 | 139.9 | 779 | 774 | 757 | 499 | 372 | 326 | 223 | 153 | 292 | 311 | 309 | 124.3 | . 419 | 5240 | 1.706 | 335 |  |
| 656 | 139.8 | 857 | 853 | 833 | 520 | 373 | 319 | 237 | 298 | 441 | 448 | 449 | 131.1 | . 373 | 5870 | 1.665 | 329 |  |
| 938 | 139.3 | 865 | 862 | 841 | 520 | 369 | 315 | 237 | 293 | 441 | 448 | 448 | 131.0 | . 364 | 6040 | 1.650 | 327 |  |
| 911 | 139.5 | 920 | 915 | 891 | 535 | 368 | 310 | 240 | 179 | 342 | 365 | 365 | 127.6 | . 337 | 6460 | 1.633 | 323 |  |
| 904 | 140.3 | 937 | 931 | 908 | 548 | 376 | 314 | 242 | 107 | 261 | 304 | 301 | 123.8 | . 336 | 6470 | 1.643 | 325 | $V$ |

(t) Stagnation temperature, $\mathrm{T}_{0}, 156 \mathrm{~K}$

| 117-729 | 158.4 | 666 | 665 | 656 | 497 | 374 | 319 | 138 | 139 | 227 | 240 | 240 | 130.1 | 0.479 | 2660 | $\mathrm{b}_{2.211}$ | 244 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 671 | 156.3 | 693 | 691 | 682 | 512 | 385 | 329 | 150 | 61 | 158 | 187 | 188 | 119.3 | . 474 | 2860 | ${ }^{b_{2.149}}$ | 274 |  |
| 751 | 155.1 | 705 | 703 | 693 | 518 | 389 | 331 | 155 | 63 | 163 | 192 | 194 | 118.0 | . 470 | 3090 | ${ }^{\text {b }} 2.116$ | 289 |  |
| 948 | 153.8 | 756 | 753 | 742 | 546 | 406 | 345 | 173 | 211 | 324 | 333 | 332 | 129.8 | . 456 | 3500 | $\mathrm{b}_{2.046}$ | 313 |  |
| 670 | 154.4 | 870 | 365 | 851 | 610 | 445 | 373 | 204 | 87 | 221 | 257 | 257 | 120.4 | . 429 | 4100 | $\mathrm{b}_{1.964}$ | 330 |  |
| 947 | 157.8 | 950 | 946 | 929 | 661 | 479 | 398 | 211 | 252 | 400 | 412 | 410 | 134.9 | . 419 | 4480 | ${ }^{\text {b }} 1.975$ | 328 | $\gamma$ |

(u) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 177 \mathrm{~K}$

| 117-506 | 179.3 | 473 | 471 | 464 | 360 | 274 | 236 | 78 | 47 | 109 | 125 | 127 | 281.8 | 0.500 | 1450 | $\mathrm{b}_{2.623}$ | 58 | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 697 | 175.0 | 590 | 587 | 579 | 447 | 338 | 290 | 97 | 59 | 140 | 161 | 162 | 269.4 | 492 | 1950 | $\mathrm{b}_{2.483}$ | 106 |  |
| 691 | 176.8 | 696 | 694 | 685 | 523 | 396 | 339 | 113 | 119 | 202 | 221 | 221 | 268.8 | . 487 | 2330 | $\mathrm{b}_{2.416}$ | 36 |  |
| 672 | 175.5 | 880 | 877 | 865 | 650 | 486 | 413 | 149 | 65 | 201 | 234 | 234 | 142.3 | . 469 | 3140 | $\mathrm{b}_{2.275}$ | 210 |  |
| 690 | 178.2 | 912 | 906 | 894 | 672 | 502 | 426 | 148 | 156 | 262 | 289 | 288 | 263.8 | . 467 | 3200 | $\mathrm{b}_{2.290}$ | 207 | $\eta$ |

(v) Stagnation temperature, $\mathrm{T}_{0}, 234 \mathrm{~K}$

| 117-726 | 234.7 | 85 | 87 | 85 | 68 | 53 | 45 | 15 | 18 | 31 | 30 | 34 | 232.3 | 0.527 | 200 | $\mathrm{b}_{3.526}$ | (c) | (a) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 725 | 234.8 | 149 | 151 | 148 | 117 | 90 | 77 | 26 | 32 | 53 | 54 | 58 | 231.5 | . 515 | 340 | $\mathrm{b}_{3.353}$ | (c) | (a) |
| 747 | 233.1 | 313 | 311 | 308 | 241 | 182 | 155 | 97 | 159 | 187 | 188 | 191 | 227.5 | . 495 | 820 | $\mathrm{b}_{3.105}$ | (c) | (a) |

## ${ }^{\mathrm{a}}$ Not applicable.

$\mathrm{b}_{\mathrm{S}_{0}}>\mathrm{S}_{\mathrm{c}}=1.813$.
$\mathrm{b}_{\mathrm{S}_{\mathrm{o}}}>\mathrm{S}_{\mathrm{c}}=1.813$.
$\mathrm{c}_{\mathrm{sat}}\left(\mathrm{S}_{\mathrm{o}}\right)<10 \mathrm{~N} / \mathrm{cm}^{2}$.

TABLE X. - DATA FOR METHANE - ELLIPTICAL NOZZLE
(a) Stagnation temperature, $T_{o}, 123 \mathrm{~K}$

| Reading | Stagnation temperature, $\mathrm{T}_{\mathrm{o}}$, K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{0}, \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | $\begin{gathered} \text { Back - } \\ \text { pressure, } \\ P_{b}, \\ N / \mathrm{cm}^{2} \end{gathered}$ | Temperature at back condition, $\mathrm{T}_{\mathrm{b}}$,K | Ratio of throat pressure at stagnation pressure, $P_{t} / P_{0}$ | Maximum mass flux,$\begin{gathered} \mathrm{G}_{\max }, \\ \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ |  | Saturation pressure at stagnation entropy, $\mathrm{P}_{\mathrm{sat}}\left(\mathrm{S}_{0}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ | Saturation pressure at stagnation temperature, $\mathrm{P}_{\text {sat }}\left(\mathrm{T}_{0}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ |  | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-1098 | 123.4 | 149 | 152 | 146 | 58 | 31 | 23 | 22 | 20 | 22 | 30 | 38 | 124.1 | 0.154 | 3030 | 5.268 | 24 | 24 |
| 1093 | 123.0 | 182 | 185 | 177 | 65 | 32 | 23 | 21 | 23 | 61 | 64 | 69 | 124.0 | 125 | 3420 | 5.254 | 23 |  |
| 969 | 123.0 | 292 | 295 | 281 | 100 | 43 | 28 | 26 | --- | 51 | 84 | 82 | 125.2 | . 094 | 4370 | 5.244 | 22 |  |
| 965 | 123.1 | 324 | 326 | 311 | 106 | 41 | 23 | 21 | --- | 38 | 72 | 79 | 124.4 | . 073 | 4690 | 5.244 | 22 | 1 |
| 966 | 121.8 | 425 | 426 | 406 | 133 | 47 | 22 | 20 | --- | 42 | 87 | 93 | 123.4 | . 052 | 5430 | 5.197 | 20 | 22 |
| 1036 | 123.9 | 478 | 478 | 455 | 153 | 51 | 24 | 21 | 50 | 200 | 205 | 208 | 125.0 | . 050 | 5710 | 5.253 | 23 | 25 |
| 967 | 121.1 | 510 | 510 | 487 | 155 | 51 | 21 | 19 | --- | 45 | 101 | 106 | 122.9 | . 042 | 5980 | 5. 169 | 19 | 21 |
| 1035 | 122.6 | 613 | 613 | 583 | 187 | 59 | 26 | 21 | 77 | 271 | 278 | 279 | 124.3 | . 042 | 6540 | 5.204 | 20 | 23 |
| 968 | 121.5 | 717 | 715 | 682 | 209 | 65 | 26 | 20 | --- | 90 | 158 | 161 | 124.0 | . 036 | 7100 | 5.164 | 19 | 21 |

(b) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 126 \mathrm{~K}$

| 117-1096 | 127.7 | 65 | 69 | 67 | 40 | 32 | 28 | 28 | 26 | 28 | 30 | 36 | 127.8 | 0.425 | 1680 | 5. 400 | 32 | 32 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1095 | 125.6 | 89 | 92 | 89 | 44 | 30 | 25 | 26 | 25 | 38 | 38 | 44 | 125.8 | 284 | 2170 | 5.338 | 28 | 28 |
| 1097 | 124.8 | 112 | 115 | 111 | 49 | 31 | 25 | 24 | 22 | 23 | 29 | 36 | 125.3 | 221 | 2520 | 5.313 | 26 | 27 |
| 1094 | 124.2 | 116 | 118 | 114 | 50 | 30 | 23 | 24 | 23 | 43 | 45 | 50 | 124.6 | . 203 | 2590 | 5.295 | 25 | 26 |
| 1038 | 127.6 | 278 | 280 | 267 | 99 | 44 | 29 | 27 | 42 | 123 | 125 | 130 | 128.2 | 105 | 4200 | 5.377 | 30 | 32 |
| 1037 | 125.8 | 358 | 360 | 342 | 121 | 46 | 26 | 24 | 44 | 154 | 157 | 161 | 126.6 | . 074 | 4870 | 5.318 | 26 | 28 |
| 1010 | 127.8 | 618 | 619 | 590 | 192 | 63 | 31 | 26 | 90 | 280 | 285 | 288 | 129.3 | . 050 | 6430 | 5.351 | 28 | 32 |
| 1009 | 126.3 | 737 | 737 | 701 | 222 | 71 | 33 | 24 | 36 | 233 | 244 | 246 | 128.6 | . 045 | 7100 | 5. 299 | 25 | 29 |

(c) Stagnation temperature, $\mathrm{T}_{0}, 133 \mathrm{~K}$

| 117-1040 | 132.8 | 160 | 162 | 155 | 74 | 47 | 39 | 34 | 34 | 39 | 48 | 56 | 133.0 | 0.242 | 2910 | 5.533 | 42 | 43 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1013 | 136.1 | 216 | 217 | 209 | 93 | 57 | 45 | 42 | 49 | 101 | 101 | 107 | 136.3 | . 207 | 3410 | 5.616 | 50 | 52 |
| 1039 | 129.6 | 220 | 222 | 212 | 85 | 44 | 32 | 29 | 34 | 35 | 87 | 92 | 130.1 | . 147 | 3640 | 5.438 | 35 | 36 |
| 1012 | 133.3 | 284 | 285 | 272 | 108 | 54 | 39 | 35 | 50 | 130 | 131 | 136 | 133.8 | . 136 | 4100 | 5.533 | 42 | 45 |
| 1011 | 131.2 | 391 | 392 | 374 | 136 | 56 | 35 | 30 | 58 | 176 | 179 | 183 | 132.1 | . 991 | 4970 | 5.465 | 37 | 40 |

(d) Stagnation temperature, $\mathrm{T}_{\mathrm{O}}, 154 \mathrm{~K}$

| Reading | Stagnation temperature, $K_{0}$, K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{\mathrm{o}},{ }_{2}^{2} \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Backpressure, $P_{b}$, $\mathrm{N} / \mathrm{cm}^{2}$ | Tempera- <br> ture at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux, $\mathrm{G}_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} S_{0^{\prime}} \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation pressure at stagnation entropy,$\begin{gathered} \mathrm{P}_{\mathrm{sat}\left(\mathrm{~S}_{0}\right)} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~T}_{0}\right) \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $P_{9}$ |  |  |  |  |  |  |  |
| 117-988 | 156.6 | 226 | 230 | 225 | 150 | 115 | 101 | 98 | 100 | 132 | 135 | 143 | 156.1 | 0.448 | 2650 | 6.164 | 135 | 139 |
| 989 | 156.4 | 283 | 288 | 280 | 171 | 130 | 112 | 90 | 95 | 133 | 142 | 148 | 156.2 | . 397 | 3200 | 6.148 | 132 | 138 |
| 987 | 155.1 | 293 | 296 | 288 | 169 | 127 | 110 | 87 | 97 | 145 | 149 | 156 | 155.1 | . 375 | 3360 | 6.111 | 125 | 131 |
| 986 | 154.0 | 352 | 354 | 343 | 181 | 127 | 107 | 80 | 105 | 181 | 184 | 190 | 154.0 | . 304 | 3930 | 6.072 | 117 | 125 |
| 990 | 155.1 | 425 | 428 | 413 | 203 | 135 | 111 | 68 | 87 | 151 | 168 | 174 | 155.3 | . 262 | 4420 | 6.090 | 120 | 131 |
| 985 | 152.8 | 427 | 428 | 413 | 196 | 125 | 100 | 74 | 118 | 222 | 226 | 232 | 153.0 | . 235 | 4550 | 6.029 | 109 | 118 |
| 991 | 154.4 | 559 | 561 | 540 | 237 | 136 | 104 | 69 | 97 | 222 | 231 | 235 | 154.7 | . 185 | 5330 | 6.051 | 113 | 127 |
| 984 | 151.5 | 573 | 574 | 553 | 232 | 126 | 91 | 68 | 145 | 293 | 299 | 303 | 151.9 | . 159 | 5560 | 5.975 | 99 | 112 |
| 992 | 153.9 | 693 | 694 | 666 | 272 | 140 | 100 | 69 | 103 | 276 | 286 | 289 | 154.4 | . 144 | 6080 | 6.020 | 107 | 124 |
| 993 | 154.3 | 747 | 747 | 718 | 288 | . 144 | 100 | 70 | 105 | 297 | 308 | 310 | 154.8 | . 134 | 6330 | 6.023 | 108 | 126 |
| 983 | 150.1 | 772 | 773 | 741 | 284 | 130 | 86 | 66 | 190 | 389 | 398 | 401 | 150.9 | . 112 | 6680 | 5.914 | 89 | 105 |

(e) Stagnation temperature, $\mathrm{T}_{0}, 167 \mathrm{~K}$

| $117-1044$ | 167.3 | 424 | 424 | 412 | 248 | 175 | 149 | 126 | 156 | 218 | 226 | 230 | 166.6 | 0.351 | 3740 | 6.413 | 195 | 211 |
| ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: | ---: |
| 1043 | 166.9 | 492 | 492 | 477 | 272 | 200 | 172 | 101 | 148 | 216 | 229 | 233 | 166.1 | .349 | 4190 | 6.387 | 189 |  |
| 1042 | 166.6 | 599 | 599 | 578 | 301 | 208 | 176 | 87 | 143 | 236 | 252 | 256 | 165.7 | .295 | 4900 | 6.357 | 181 |  |

(f) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 184 \mathrm{~K}$

| 117-1017 | 182.1 | 501 | 500 | 489 | 327 | 258 | 238 | 221 | 177 | 226 | 241 | 245 | 169.3 | 0.475 | 3420 | 6.786 | 315 | 351 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1023 | 184.1 | 555 | 555 | 542 | 359 | 281 | 257 | 232 | 117 | 240 | 254 | 255 | 170.6 | . 464 | 3620 | 6.818 | 326 | 374 |
| 1025 | 185.5 | 576 | 575 | 562 | 377 | 300 | 275 | 241 | 190 | 243 | 262 | 265 | 171.9 | . 477 | 3620 | 6.849 | 337 | 391 |
| 1016 | 182.3 | 628 | 627 | 611 | 369 | 263 | 225 | 194 | 198 | 301 | 310 | 314 | 177.3 | . 358 | 4230 | 6.737 | 298 | 353 |
| 1024 | 185.6 | 737 | 735 | 714 | 423 | 305 | 261 | 216 | 131 | 285 | 307 | 308 | 176.7 | . 355 | 4620 | 6.781 | 313 | 393 |
| 1015 | 182.3 | 746 | 746 | 723 | 411 | 293 | 252 | 168 | 166 | 304 | 320 | 324 | 178.3 | . 337 | 4860 | 6.697 | 284 | 353 |

(g) Stagnation temperature, $\mathrm{T}_{0}, 195 \mathrm{~K}$

| 117-996 | 194.4 | 625 | 627 | 618 | 476 | 425 | 406 | 223 | 199 | 293 | 303 | 307 | 176.5 | 0.649 | 2910 | 7.158 | 429 | --- |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 995 | 195.6 | 724 | 725 | 711 | 498 | 417 | 388 | 259 | 242 | 356 | 366 | 369 | 182.7 | . 536 | 3630 | 7. 108 | 417 | --- |
| 994 | 194.7 | 747 | 748 | 733 | 498 | 405 | 373 | 268 | 258 | 374 | 384 | 388 | 184.3 | . 499 | 3900 | 7.060 | 405 | --- |

(h) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 200 \mathrm{~K}$

| 117-1101 | 198.2 | 690 | 690 | 680 | 519 | 454 | 436 | 236 | 161 | 211 | 238 | 242 | 169.0 | 0.632 | 2980 | 7.247 | 444 | --- |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1100 | 200.0 | 802 | 800 | 785 | 551 | 452 | 424 | 268 | 194 | 255 | 288 | 292 | 174.8 | 529 | 3690 | 7.199 | 436 | --- |
| 1099 | 201.2 | 925 | 920 | 901 | 585 | 449 | 410 | 294 | 219 | 295 | 335 | 338 | 179.7 | . 443 | 4390 | 7.150 | 427 | --- |

TABLE XI. - DATA FOR HYDROGEN - ELLIPTICAL NOZZLE
(a) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 28 \mathrm{~K}$

| Reading | Stagnation temperature, $T_{0}$, K | Stagnation pressure,$\begin{gathered} \mathrm{P}_{\mathrm{O}}{ }^{\prime} 2 \\ \mathrm{~N} / \mathrm{cm}^{2} \end{gathered}$ | Pressures at stations 1 to $9, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Back- <br> pressure, $\begin{gathered} \mathrm{P}_{\mathrm{b}} \\ \mathrm{~N} / \mathrm{cm} 2 \end{gathered}$ | Temperature at back condition, $T_{b}$, K | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | $\begin{aligned} & \text { Maximum } \\ & \text { mass flux }, \\ & G_{\max }, \\ & \mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{aligned}$ | Stagnation entropy ,$\begin{gathered} \mathrm{S}_{0^{\prime}} \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | $\begin{array}{cc} \text { Saturation } & \text { Saturation } \\ \text { pressure } & \text { pressure } \\ \text { at stagna- } & \text { at stagna- } \\ \text { tion en- } & \text { tion tem- } \\ \text { tropy, } & \text { perature, } \\ \mathrm{P}_{\text {sat }}\left(\mathrm{S}_{0}\right), & \mathrm{P}_{\text {sat }}\left(\mathrm{T}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} & \mathrm{~N} / \mathrm{cm}^{2} \end{array}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ |  | $\mathrm{P}_{5}$ |  | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ |  |  |  |  |  |  |  |
| 117-1199 | 27.8 | 293 | 295 | 284 | 118 | 58 | 42 | 18 | 26 | 65 | 82 | 87 | 28.4 | 0,143 | 1700 | 10.80 | 39 | 57 |
| 1200 | 27.2 | 340 | 231 | 327 | 127 | 57 | 38 | 17 | 24 | 59 | 85 | 90 | 28.2 | . 113 | 1870 | 10.39 | 33 | 51 |
| 1212 | 28.7 | 431 | 430 | 413 | 161 | 68 | 45 | 20 | 37 | 127 | 136 | 138 | 29.6 | . 104 | 2090 | 10.80 | 39 | 66 |
| 1210 | 28.7 | 471 | 470 | 452 | 172 | 69 | 46 | 20 | 33 | 103 | 128 | 129 | 29.7 | . 098 | 2210 | 10.70 | 38 | 66 |
| 1222 | 28.1 | 582 | 579 | 557 | 206 | 72 | 43 | 20 | 55 | 209 | 216 | 217 | 29.9 | . 074 | 2500 | 10.19 | 31 | 60 |
| 1216 | 28.6 | 589 | 587 | 564 | 210 | 76 | 44 | 20 | 36 | 133 | 160 | 161 | 30.2 | . 075 | 2510 | 10.38 | 33 | 65 |

(b) Stagnation temperature, $\mathrm{T}_{\mathrm{o}}, 30 \mathrm{~K}$

| 117-1197 | 30.7 | 129 | 133 | 128 | 75 | 51 | 47 | 43 | 41 | 62 | 64 | 69 | 28.7 | 0.364 | 890 | 13.27 | 83 | 92 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1198 | 29.0 | 219 | 222 | 213 | 101 | 62 | 51 | 18 | 34 | 70 | 76 | 82 | 29.0 | 232 | 1370 | 11.66 | 53 | 70 |
| 1207 | 30.3 | 288 | 288 | 277 | 126 | 72 | 63 | 24 | 64 | 133 | 135 | 137 | 30.4 | . 220 | 1570 | 12.02 | 59 | 86 |
| 1213 | 29.5 | 356 | 356 | 342 | 142 | 69 | 49 | 21 | 41 | 118 | 124 | 126 | 29.8 | . 139 | 1840 | 11.38 | 48 | 76 |
| 1206 | 29.5 | 360 | 360 | 346 | 145 | 69 | 67 | 22 | 49 | 137 | 141 | 143 | 30.0 | . 185 | 1850 | 11.37 | 48 | 76 |
| 1219 | 30.6 | 370 | 370 | 356 | 152 | 77 | 57 | 22 | 37 | 98 | 113 | 115 | 30.5 | . 154 | 1830 | 11.84 | 56 | 90 |
| 1211 | 29.1 | 422 | 421 | 405 | 160 | 69 | 46 | 20 | 34 | 100 | 120 | 122 | 29.8 | . 110 | 2050 | 11.00 | 43 | 71 |
| 1217 | 30.3 | 421 | 420 | 405 | 166 | 77 | 54 | 22 | 43 | 126 | 137 | 138 | 30.6 | . 128 | 1990 | 11.53 | 51 | 86 |
| 1223 | 29.8 | 425 | 425 | 409 | 165 | 74 | 51 | 22 | 56 | 163 | 169 | 170 | 30.3 | . 119 | 2020 | 11.30 | 47 | 80 |

(c) Stagnation temperature, $\mathrm{T}_{0}, 32 \mathrm{~K}$

| 117-1215 | 32.3 | 167 | 168 | 164 | 106 | 83 | 75 | 59 | 69 | 95 | 95 | 98 | 31.1 | 0.448 | 900 | 14.10 | 97 | 117 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1214 | 31.0 | 223 | 224 | 217 | 112 | 74 | 62 | 35 | 67 | 111 | 113 | 115 | 30.7 | . 276 | 1270 | 12.72 | 72 | 96 |
| 1221 | 32.2 | 221 | 222 | 215 | 119 | 80 | 66 | 50 | 48 | 87 | 91 | 93 | 30.7 | . 298 | 1190 | 13.47 | 86 | 115 |
| 1224 | 31.3 | 281 | 285 | 272 | 131 | 79 | 65 | 32 | 75 | 138 | 141 | 143 | 31.0 | . 231 | 1500 | 12.56 | 69 | 101 |
| 1218 | 32.3 | 285 | 284 | 275 | 140 | 89 | 74 | 39 | 73 | 127 | 130 | 132 | 31.6 | . 260 | 1430 | 13.09 | 79 | 117 |
| 1220 | 31.3 | 290 | 291 | 281 | 132 | 80 | 65 | 22 | 45 | 98 | 105 | 107 | 30.7 | . 225 | 1520 | 12.52 | 69 | 101 |
| 1201 | 32.2 | 294 | 295 | 286 | 152 | 96 | 74 | 33 | 36 | 76 | 90 | 95 | 30.1 | . 252 | 1420 | 12.98 | 77 | 115 |

TABLE XII. - DATA FOR NITROGEN WITH SPECIAL EMPHASIS ON THERMODYNAMIC

CRITICAL REGION - ELLIPTICAL NOZZLE

| Reading | Stagnation temperature, $\mathrm{T}_{\mathrm{o}}$, K | Pressures at stations 1 to 9, N/ $\mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  | Ratio of throat pressure to stagnation pressure, $\mathbf{P}_{\mathrm{t}} / \mathrm{P}_{\mathrm{o}}$ | Maximum mass fiux, $G_{\max }$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | Stagnation entropy,$\begin{gathered} \mathbf{S}_{\mathbf{o}}, \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Saturation <br> pressure <br> at stagna - <br> tion en- <br> tropy, $\begin{gathered} \mathrm{P}_{\mathrm{sat}}\left(\mathrm{~S}_{0}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature, $\mathrm{P}_{\text {sat }}\left(\mathrm{T}_{\mathrm{o}}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathbf{P}_{6}$ | $\mathbf{P}_{7}$ | $\mathrm{P}_{8}$ | $P_{9}$ |  |  |  |  |  |
| 416-79 | 118.3 | 389 | 381 | 220 | 143 | 121 | 98 | 87 | 100 | 117 | 0.311 | 4850 | 1.314 | 205 | 231 |
| 80 | 118.6 | 391 | 382 | 221 | 144 | 123 | 103 | 88 | 148 | 158 | . 315 | 4890 | 1.321 | 208 | 234 |
| 81 | 119.4 | 421 | 412 | 242 | 162 | 136 | 112 | 89 | 101 | 119 | . 323 | 5030 | 1.332 | 213 | 244 |
| 82 | 119.7 | 421 | 412 | 243 | 163 | 138 | 120 | 89 | 146 | 160 | . 328 | 5000 | 1.340 | 216 | 248 |
| 102 | 120.0 | 290 | 288 | 210 | 183 | 175 | 161 | 62 | 72 | 83 | . 603 | 3190 | 1.399 | 242 | 251 |
| 103 | 120.1 | 291 | 289 | 212 | 184 | 177 | 162 | 63 | 117 | 125 | . 608 | 3160 | 1.402 | 243 | 253 |
| 108 | 121.5 | 389 | 382 | 249 | 191 | 174 | 163 | 79 | 84 | 107 | . 447 | 4350 | 1.399 | 242 | 271 |
| 109 | 121.8 | 391 | 384 | 252 | 196 | 178 | 166 | 79 | 156 | 167 | . 455 | 4290 | 1.407 | 245 | 275 |
| 84 | 125.3 | 326 | 325 | 275 | 216 | 193 | 89 | 34 | 51 | 61 | . 592 | 1980 | 1.644 | 325 | 326 |
| 85 | 125.3 | 327 | 327 | 276 | 216 | 194 | 91 | 61 | 101 | 109 | . 593 | 2030 | 1.638 | 324 | 326 |
| 48 | 125.5 | 340 | 339 | 284 | 221 | 199 | 96 | 36 | 54 | 66 | . 585 | 2180 | 1.612 | 318 | 328 |
| 49 | 125.4 | 340 | 339 | 286 | 221 | 200 | 96 | 64 | 105 | 111 | . 588 | 2210 | 1.603 | 316 | 327 |
| 87 | 126.2 | 358 | 357 | 309 | 235 | 212 | 102 | 38 | 55 | 67 | . 592 | 2260 | 1.623 | 321 | 340 |
| 88 | 126.2 | 358 | 358 | 309 | 236 | 213 | 102 | 72 | 108 | 116 | . 595 | 2260 | 1.623 | 321 |  |
| 32 | 126.2 | 342 | 339 | 286 | 221 | 199 | 89 | 33 | 50 | 63 | . 582 | 2050 | 1.702 | 335 |  |
| 33 | 126.2 | 344 | 341 | 287 | 222 | 200 | 89 | 52 | 60 | 103 | . 581 | 2040 | 1.682 | 332 |  |
| 93 | 126.2 | 342 | 342 | 287 | 225 | 202 | 92 | 34 | 52 | 63 | . 591 | 2010 | 1.702 | 335 |  |
| 94 | 126.2 | 343 | 344 | 288 |  | 202 | 91 | 69 | 115 | 121 | . 589 | 2000 | 1.691 | 333 | $\dagger$ |
| 95 | 126.3 | 344 | 344 | 289 |  | 203 | 91 | 112 | 157 | 159 | . 590 | 2000 | 1.706 | 335 | 342 |
| 96 | 126.3 | 345 | 345 | 289 | $\dagger$ | 203 | 94 | 151 | 192 | 192 | . 588 | 1990 | 1.694 | 334 | 342 |
| 97 | 126.4 | 345 | 345 | 290 | 226 | 203 | 149 | 202 | 227 | 227 | . 588 | 1980 | 1.725 | 337 | (a) |
| 98 | 126.4 | 346 | 345 | 291 | 225 | 203 | 210 | 250 | 271 | 270 | . 587 | 1970 | 1.709 | 336 |  |
| 26 | 126.7 | 367 | 363 | 313 | 240 | 217 | 103 | 38 | 57 | 71 | . 591 | 2280 | 1.640 | 325 |  |
| 27 | 126.7 | 370 | 367 | 317 | 242 | 218 | 104 | 76 | 105 | 115 | . 589 | 2330 | 1.632 | 323 |  |
| 91 | 127.9 | 422 | 420 | 340 | 314 | 302 | 138 | 54 | 71 | 88 | . 716 | 3000 | 1.611 | 318 |  |
| 92 | 128.0 | 424 | 421 | 341 | 314 | 303 | 138 | 55 | 121 | 134 | 715 | 2980 | 1.613 | 318 |  |
| 54 | 128.0 | 396 | 394 | 336 | 264 | 240 | 113 | 42 | 60 | 76 | . 606 | 2440 | 1.666 | 330 |  |
| 55 | 128.0 | 393 | 391 | 334 | 260 | 236 | 111 | 45 | 97 | 109 | . 601 | 2390 | 1.674 | 331 |  |
| 42 | 130.3 | 421 | 418 | 355 | 282 | 251 | 114 | 42 | --- | 77 | . 596 | 2300 | 1.778 | 341 |  |
| 43 | 130.5 | 423 | 420 | 356 | 285 | 253 | 114 | 60 | 120 | 131 | . 598 | 2320 | 1.787 | 341 | 1 |

[^2]TABLE XIII. - DATA FOR NITROGEN FROM INITIAL FLOW RATE STUDIES
ELLIPTICAL NOZZLE

| Reading | Stagnation temperature, $\mathrm{T}_{\mathrm{o}}$, K | Pressures at stations 1,5, and 9, $\mathrm{N} / \mathrm{cm}^{2}$ |  |  | Ratio of throat pressure to stagnation pressure, $P_{t} / P_{o}$ | Maximum mass flux, $\mathrm{G}_{\text {max }}$,$\mathrm{g} / \mathrm{cm}^{2} \cdot \mathrm{sec}$ | $\begin{gathered} \text { Stagnation } \\ \text { entropy, } \\ \mathrm{S}_{\mathbf{o}^{\prime}} \\ \mathrm{J} / \mathbf{g} \cdot \mathbf{K} \end{gathered}$ | Saturation pressure at stagnation entropy, $\mathbf{P}_{\text {sat }}\left(\mathrm{S}_{0}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ | Saturation pressure at stagnation temperature,$\begin{gathered} \mathbf{P}_{\mathrm{sat}}{ }^{\left(\mathrm{T}_{0}\right)}, \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{5}$ | $\mathbf{P}_{9}$ |  |  |  |  |  |
| 616-352 | 89.9 | 113 | 30 | 38 | 0.265 | 3340 | 0.719 | 35 | 36 |
| 338 | 89.9 | 152 | 30 | 43 | . 197 | 4050 | . 716 | 34 |  |
| 356 | 89.9 | 202 | 28 | 54 | . 139 | 4890 | . 711 | 34 |  |
| 367 | 90.0 | 253 | 29 | 66 | . 115 | 5520 | . 709 | 33 |  |
| 298 | 90.1 | 294 | 29 | 43 | . 009 | 5890 | . 708 | 33 |  |
| 375 | 90.0 | 323 | 29 | 72 | . 090 | 6280 | . 703 | 33 |  |
| 409 | 90.0 | 344 | 28 | 80 | . 081 | 6530 | . 701 | 32 | 1 |
| 312 | 90.4 | 384 | 30 | 42 | . 078 | 6850 | . 707 | 33 | 37 |
| 393 | 90.0 | 425 | 28 | 88 | . 066 | 7330 | . 694 | 32 | 36 |
| 340 | 99.9 | 154 | 58 | 56 | . 377 | 3340 | . 936 | 75 | 77 |
| 358 | 99.8 | 202 | 63 | 67 | . 312 | 4140 | . 928 | 73 | 77 |
| 370 | 99.9 | 253 | 62 | 76 | . 245 | 4880 | . 924 | 72 | 77 |
| 299 | 100.1 | 295 | 62 | 100 | . 210 | 5290 | . 924 | 72 | 78 |
| 377 | 100.1 | 319 | 61 | 95 | . 191 | 5660 | . 921 | 71 |  |
| 403 | 100.1 | 344 | 61 | 104 | . 177 | 5920 | . 918 | 71 |  |
| 314 | 100.1 | 384 | 60 | 79 | . 156 | 6320 | . 914 | 70 |  |
| 396 | 100.1 | 426 | 59 | 78 | . 138 | 6770 | . 909 | 68 | $\dagger$ |
| 342 | 109.1 | 153 | 66 | 62 | . 431 | 2810 | 1.167 | 145 | 146 |
| 372 | 110.0 | 253 | 81 | 88 | . 320 | 4200 | 1.148 | 139 | 147 |
| 302 | 110.0 | 297 | 102 | 93 | . 343 | 4550 | 1.140 | 136 | 147 |
| 380 | 110.1 | 324 | 107 | 99 | . 330 | 4840 | 1.138 | 135 | 148 |
| 406 | 110.0 | 346 | 110 | 100 | . 318 | 5110 | 1.132 | 133 | 147 |
| 316 | 110.0 | 386 | 113 | 105 | . 293 | 5480 | 1.125 | 130 | 147 |
| 398 | 110.0 | 426 | 111 | 109 | . 261 | 5970 | 1.118 | 128 | 147 |
| 374 | 120.1 | 255 | 189 | 70 | . 741 | 2470 | 1.423 | 252 | 253 |
| 303 | 120.0 | 295 | 170 | 135 | . 576 | 3340 | 1.396 | 241 | 251 |
| 381 | 120.0 | 326 | 152 | 145 | . 466 | 4010 | 1.382 | 234 | 251 |
| 407 | 120.0 | 346 | 146 | 153 | . 422 | 4260 | 1.374 | 231 | 251 |
| 317 | 120.0 | 385 | 138 | 152 | . 358 | 4640 | 1.359 | 224 | 251 |
| 387 | 120.0 | 425 | 138 | 166 | . 325 | 5040 | 1.346 | 219 | 251 |
| 413 | 125.0 | 345 | 249 | 112 | . 722 | 2370 | 1.565 | 305 | 321 |
| 319 | 125.0 | 387 | 255 | 117 | . 659 | 3250 | 1.519 | 290 | 321 |
| 399 | 125.1 | 427 | 234 | 166 | . 548 | 4040 | 1.495 | 281 | 323 |

TABLE XIV. - CHOKED AND UNCHOKED PRESSURE DISTRIBUTION IN $3.5^{\circ}$ CONICAL NOZZLE

| Reading | Stagna- <br> tion <br> temper- <br> ature, $T_{0}$ $\mathrm{K}$ | $\begin{array}{\|c\|} \hline \text { Stagna- } \\ \text { tion } \\ \text { pres- } \\ \text { sure, } \\ P_{0}, \\ N / \mathrm{cm}^{2} \end{array}$ | Pressures at stations 1 to $15, \mathrm{~N} / \mathrm{cm}^{2}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Backpressure,$\begin{gathered} P_{b^{\prime}}^{\prime} \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Tempera- <br> ture at back condition, $\mathrm{T}_{\mathrm{b}}$, K | Ratio of <br> throat <br> pres- <br> sure to <br> stagna- <br> tion <br> pressure, $P_{t} / P_{0}$ | Mass flux, G$\mathrm{g} / \mathrm{cm}^{2}$ . sec | Stagna- <br> tion en- <br> tropy, $\begin{gathered} \mathrm{S}_{\mathbf{o}}, \\ \mathrm{J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | $\begin{gathered} \begin{array}{c} \text { Satura- } \\ \text { tion } \end{array} \\ \text { pressure } \\ \text { at stagna- } \\ \text { tion en- } \\ \text { tropy, } \\ \mathrm{P}_{\text {sat }}\left(\mathrm{S}_{\mathrm{o}}\right), \\ \mathrm{N} / \mathrm{cm}^{2} \end{gathered}$ | Saturation pressure at stagnation temperature, $P_{s a t}\left(T_{0}\right)$, $\mathrm{N} / \mathrm{cm}^{2}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | $\mathrm{P}_{1}$ | $\mathrm{P}_{2}$ | $\mathrm{P}_{3}$ | $\mathrm{P}_{4}$ | $\mathrm{P}_{5}$ | $\mathrm{P}_{6}$ | $\mathrm{P}_{7}$ | $\mathrm{P}_{8}$ | $\mathrm{P}_{9}$ | $\mathrm{P}_{10}$ | $\mathrm{P}_{11}$ | $\mathrm{P}_{12} \mathrm{P}^{\text {a }}$ | $\mathrm{P}_{13}$ | ${ }_{14}$ | ${ }^{1} 15$ |  |  |  |  |  |  |  |
| 101-1331 | 109.1 | 471 | 472 | 471 | 471 | 451 | 421 | 385 | 342 | 318 | 309 | 308 | 370 | 379 | 385 | 398 | 409 | 412 | 109.3 | 0.654 | 4370 | 1.092 | 119 | 139 |
| 1332 | 109.4 | 465 | 466 | 466 ! | 465 | 431 | 379 | 317 | 244 | 199 | 184 | 182 | 288 | 304 | 315 | 336 | 357 | 360 | 109.5 | . 391 | 5650 | 1.099 | 122 | 142 |
| 1333 | 109.9 | 463 | '463 | 463 ' | 461 | : 422 ' | 360' | 286 | 200 | 145 | 127 | 126 | 99 | 109 | 117 | 132 | 163 | 177 | 109.6 | . 269 | 6160 | 1.111 | 125 | 146 |
| 1334 | 111.0 | 463 | 463 | 462 | 461 | 422 | 362 | 289 | 204 | 150 | 133 | 130 | 50 | 39 | 31 | 45 | 51 | 54 | 95.0 | . 281 | 6050 | 1.133 | 133 | 155 |
| 1350 | 109.7 | 227 | 227 | 227 | 227 | 218 | 204 | 188 | 168 | 156 | 152 | 152 | 181. | 185 | 188 | 194 | 199 | 200 | 109.7 | . 669 | 2920 | 1.147 | 138 | 144 |
| 1351 | 109.9 | 226 | 225 | 226 | 226 | 214 | 195, | 173 | 147 | 130 | 124 | 116 | 1301 | \| 134 | 139 | 150 | 161 | 162 | 109.7 | . 511 | 3340 | 1.152 | 140 | 146 |
| 1352 | 110.2 | 226 | , 225 | - 226 | 225 | , 213 | 1951 | 173 | 146 ' | 128 | 121 | 109 | 112 | 114 | 117 | 122 | 131 | 136 | 109.0 | . 483 | 3340 | 1.160 | 143 | 149 |
| 1353 | 110.5 | 225 | 225 | 226 | 225 | 213 | 195 | 172 | 146 | 127 | 119 | 106 | 102 | 104 | 106 | 110 | 116 | 122 | 106.8 | . 469 | 3330 | 1.166 | 145 | 151 |
| 1354 | 110.9 | 225 | 225 | : 225 | 225 | 213 | 194 ! | 172 | 145 | 125 | 117 | 100 | 55 | 58 | 60 | 65 | 69 | 72 | 98.8 | . 446 | 3350 | 1.176 | 149 | 155 |
| 1344 | 283.1 | 353 | 353 | 354 | 353 | 342 | 324 | 299 | 2597 | 218 | 208 | 190 | 42 | 31 | 20 | 8 | 13 | 18 | 269.9 | . 537 | 879 | 3.282 |  |  |
| 1345 | 277.9 | 354 | 353 | 353 | , 353 | \| 342 | 324 | 299 | 259 | 219 | 207 | 190 | 42 | 31 | 27 | 45 | 56 | 66 | 272.5 | . 537 | 824 | 3.262 |  |  |

TABLE XV. - CORRESPONDING-STATES NORMALIZING PARAMETERS

| Fluid | Critical <br> pressure, $\mathrm{P}_{\mathrm{c}}{ }^{\prime}{ }_{2}$ $\mathrm{~N} / \mathrm{cm}^{2}$ | Critical temperature, $\mathrm{T}_{\mathrm{c}}$, K | Critical density, $\stackrel{\rho_{\mathrm{c}}^{\prime}}{\mathrm{g} / \mathrm{cm}^{3}}$ | Critical <br> entropy, $\begin{gathered} S_{c}, \\ \mathrm{~J} / \mathrm{g} \cdot \mathrm{~K} \end{gathered}$ | Compressibility factor, $\mathrm{Z}_{\mathrm{c}}$ | $\begin{gathered} \text { Mass flux } \\ \mathrm{G}^{*}, \\ \mathrm{~g} / \mathrm{cm}^{2} \cdot \mathrm{sec} \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Nitrogen | 341.7 | 126.3 | 0.3105 | 1.813 | 0.2937 | 6010.4 |
| Methane | 462.7 | 190.8 | . 162 | 4.214 | . 2889 | 5093.7 |
| Hydrogen | 129.3 | 32.98 | . 03143 | 17.773 | . 3023 | 1158 |



Figure 1. - Flow system schematic for nitrogen two-phase-choked-flow rig (110liter capacity).


Figure 2. - Flow system schematic for liquid-hydrogen and -methane choked-flow rig (375-liter capacity).


Figure 3. - Axisymmetric converging-diverging nozzle with conical convergence and divergence. Throat area, $A_{\mathrm{t}}, 0.0993 \mathrm{~cm}^{2}$.


Figure 4. - Two-dimensional converging-diverging nozzle. Convergence at $7^{0}$ and divergence at $3.5^{\circ}$; throat area, $A_{t}, 0.1104 \mathrm{~cm}^{2}$.


Figure 5. - Axisymmetric converging-diverging nozzle with $2: 1$ elliptical convergence. Throat area, $A_{t}, 0.0676 \mathrm{~cm}^{2}$.


Figure 6. - Inlet and outlet mixing chambers for conical and two-dimensional nozzles.


Figure 7. - Inlet mixing (stagnation) chamber for elliptical nozzle.


Figure 8. - Flowmeter used in two-phase-critical-flow rig. (All dimensions are in cm.)


Figure 9. - Range of stagnation conditions of experiments.


Figure 10. - Gas profiles in the four nozzles of this experiment.

(b) Stagnation pressure, $\mathrm{P}_{0}, 226 \mathrm{~N} / \mathrm{cm}^{2}$.

Figure 11 . - Choked and unchoked nitrogen axial pressure profiles in $3.5^{\circ}$ conical nozzle. Stagnation temperature, $\mathrm{T}_{0}, 110 \mathrm{~K}$.


Figure 12. - Nitrogen axial pressure profiles at various stagnation pressures in $3.5^{\circ}$ conical nozzle.


Figure 13. - Nitrogen pressure profiles in two conical nozzles at high stagnation pressure. Stagnation temperature, $\mathrm{T}_{0}, 119.2 \mathrm{~K}$; saturation pressure at stagnation entropy, $\mathrm{P}_{\text {sat }}\left(\mathrm{S}_{0}\right), 190 \mathrm{~N} / \mathrm{cm}^{2}$.


Figure 14. - Nitrogen axial pressure profiles in two conical nozzles at low stagnation pressure. Stagnation temperature, $\mathrm{T}_{0}, 119.2 \mathrm{~K}$.


Figure 15. - Comparison of axial pressure profiles in all four nozzles with nitrogen profile.


Figure 16. - Nitrogen axial pressure profiles at various stagnation temperatures in $7^{0}$ conical nozzle.


Figure 17. - Choked flow rates for subcooled nitrogen in $7^{\circ}$ conical nozzle.


Figure 18. - Throat-stagnation pressure ratios for choked flow of subcooled nitrogen in $7^{0}$ conical nozzle.


Figure 19. - Choked flow rates for subcooled nitrogen in two-dimensional nozzle.


Figure 20. - Throat-stagnation pressure ratios for choked flow of subcooled nitrogen in two-dimensional nozzle.


Figure 21. - Choked flow rates for s,ubcooled nitrogen in $3.5^{\circ}$ conical nozzle.


Figure 22. - Throat-stagnation pressure ratios for choked flow of subcooled nitrogen in $3.5^{\circ}$ conical nozzle.


Figure 23. - Choked flow rates for subcooled nitrogen in elliptical nozzle.


Figure 24. - Throat-stagnation pressure ratios for choked flow of subcooled nitrogen in elliptical nozzle.


Figure 25. - Summary of choked flow rates of subcooled nitrogen in the four nozzles of this experiment.


Figure 26. - Summary of throat-stagnation pressure ratios for choked flow of subcooled nitrogen in the four nozzles of this experiment.


Figure 27. - Throat pressure for choked flow of subcooled nitrogen at stagnation temperature of $119.3 \pm 0.2 \mathrm{~K}-3.5^{\circ}$ conical nozzle.


Figure 28. - Throat pressure for choked flow of subcooled nitrogen at stagnation temperature of $119.3 \pm 0.3 \mathrm{~K}$ - two conical nozzles.


Figure 29. - Throat pressure for choked flow of subcooled nitrogen at stagnation temperature of $119.3 \pm 0.3 \mathrm{~K}$ - all four nozzles.


Figure 30. - Comparison of data with theory for two-phase choked flow rates.


Figure 31. - Comparison of data with theory for two-phase-choked-flow throat-stagnation pressure ratios.


Figure 32. - Comparison of choked flow rates for nitrogen, hydrogen, and methane on a corresponding-states basis.


Figure 33. - Comparison of throat-stagnation pressure ratios for nitrogen, hydrogen, and methane on a corresponding-states basis.


[^3]National Aeronautics and Space Administration

Washington, D.C.
20546
Official Business
Penalty for Private Use, $\$ 300$

## THIRD-CLASS BULK RATE

Postage and Fees Paid National Aeronautics and Space Administration NASA-451

101 10. D, 062979 S 00903 DS DEPT OF TAE AIR FORCE AF GEAPONS LABORATORY ATTN: TECHNICAL LIBRARY (SDL) KIPTLAND AFB NM 87117


[^0]:    $\mathrm{a}_{\text {Gas checks outside of tolerance. }}$
    ${ }^{\mathrm{b}}$ Readings appear to be in error but data record offers no reason as to why they should be .
    ${ }^{c}$ Flow rates based on downstream flowmeters.
    ${ }^{\mathrm{d}}$ Transducers not zero adjusted and flow rates based on downstream flowmeters.

[^1]:    ${ }^{\text {a }}$ Not applicable.

[^2]:    ${ }^{\mathrm{a}}$ Not applicable.

[^3]:    - For sale by the Natıonal Technical Information Service. Springfield. Virginia 22161

