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Exploring the influence of micro-structure on the mechanical properties and crack bridging mechanisms of fibrous tufts

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ABSTRACT

A constitutive model for tufts bridging a mode I delamination is presented. The tuft is modelled as a rod, laterally supported by an elastic medium and clamped at both ends. A fracture mechanics approach is introduced to describe the progressive debonding of the tuft from the embedding laminate. The debonding model requires the identification of stiffness, strength and toughness properties, which depend both on the laminate/tuft architecture and the constituent materials. Such identification is carried out via experimental data obtained from tensile tests on single tufts inserted in a pre-delaminated non-crimp fabric composite. The experimental results are complemented by micro-scale finite element analysis. The mode I bridging law obtained from the constitutive model is implemented into a meso-scale cohesive zone formulation. This formulation is applied to predict the response to delamination of tufted Double Cantilever Beam (DCB) coupons. The cohesive zone approach is validated by means of experimental data from DCB tests. It is shown that the proposed micro- to meso-scale modelling approach yields results in good agreement with the experiments.

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1. Introduction

Through-the-thickness reinforcement (TTR) is applied to 2dimensional composites in order to control and suppress delamination. Most common TTR methods include *Z*-pinning [1], stitching [2] and tufting [3]. Tufting is the most recent among them and is performed by inserting carbon, glass or aramid threads through the thickness of a dry preform by means of a single needle. Neighbouring tufts are interconnected to each other by a seam on one side of the preform and form thread loops on the other. Once resin infused, tufts become integral parts of the preform architecture, making it locally 3-dimensional. Despite the proved potential of tufts to counteract the propagation of delamination in composite parts [4], a complete study of their crack bridging behaviour is not available in the open literature.

The aim of this paper is to identify and describe the influence of micro-structure on the mode I crack bridging response of tufts and use the observations made at the micro-scale as the basis for the development of a multi-scale modelling framework for tufted composites.

An analytical micro-mechanical model is proposed to simulate the mechanical response of tufts embedded in mode I delaminating composites. The governing equations and assumptions of the model are supported by experimental results obtained for singletuft coupons, complemented with observations of the tuft architecture, morphology and failure mode. The suitability of this model for the prediction of the mechanical behaviour of bridged interfaces has been assessed via its implementation into the finite element model of a DCB coupon. A cohesive zone approach [5] has been adopted for this purpose and experimental data have been used to validate the overall multi-scale modelling strategy presented.

2. Bridging mechanisms of fibrous tufts

2.1. Single-tuft tests

A set of pre-delaminated single-tuft coupons has been tested under mode I conditions in order to derive the bridging law of the tuft, i.e. the relation between the relative displacement of the surfaces of a bridged crack and the force exerted by the tuft to counteract it [6,7]. The specimens were made of four layers of biaxial carbon Non-Crimp Fabric (NCF) with an areal weight of 1010

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2

C. Osmiani et al./Composites: Part A xxx (2016) xxx-xxx

2.2. Tuft morphology

 g/m^2 , stacked in a symmetric $[(0/90)_s]_2$ layup. The stack was separated at the mid-plane by a thin release film. Each 0°/90° layer contained equally arranged 24k HTS carbon fibre tows from Tenax, held together by non-structural stitching. Each coupon was tufted with commercially available 2k HTA40 carbon fibre sewing thread, having a dry cross-section area of 0.077 mm². Tufts were inserted orthogonally to the release film. After insertion, each tuft featured a free loop end 3–5 mm long. The tufted preform was injected with aerospace grade epoxy resin (MVR444, Advanced Composites Group) using a Vacuum Assisted Resin Transfer Moulding (VARTM) process. Injection was carried out at 70 °C and 1 bar pressure, followed by cure at 160 °C for 90 min at 4 bar. The cured panel was subjected to post-cure in the oven at 180 °C for 120 min. The final thickness was 4 (± 0.01) mm, with resulting global fibre volume fraction of 56.5%. The single-tuft coupons, with dimensions $20 \text{ mm} \times 20 \text{ mm} \times 4 \text{ mm}$, were tested in out-of-plane tension. under displacement control, at a cross-head speed of 0.25 mm/ min. A Digital Image Correlation (DIC) system was used to monitor the relative opening of the testing fixtures. Testing conditions are illustrated in Fig. 1.



Fig. 1. Mode I test on single-tuft specimen. The test is carried out in displacement control with an Instron 5500R and a 5 kN load cell. The arrows at the fours corners of the T-tabs identify the monitored displacements. Two cameras, one on the front and one at the back of the specimen, have been used at this purpose. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

Micrographic analysis was carried out to assess the post-cure tuft morphology, as shown in Fig. 2a. The insertion of tufts in a preform causes a local disruption of the in-plane fibre architecture, resulting in the formation of resin-rich regions around the through-thickness reinforcement. This is consistent with what has been reported for other TTR types in the open literature [7,8]. The cross section of the tuft is modelled by both the fabric architecture and the preform layup. Micrographs have revealed resin-rich regions characterised by maximum and minimum diameters of 5.6 mm (Coefficient Of Variation (COV) = 14.3%) and 0.55 mm (COV = 14.5%), respectively. The average impregnated cross-sectional area of the tuft, measured at 25% and 75% thickness of the samples, was 0.27 mm^2 (COV = 11%). Sectioning of the specimens has shown further that tufts are characterised by curved profiles and a random arrangement of their constituting thread segments, as in Fig. 2b. Such complex inherent features render a topological definition of tufts very difficult, and help explain the large experimental scatter in the derived bridging laws, as in Fig. 3a.





(b)

Fig. 2. In-plane (a) and out-of-plane (b) tuft morphology in a NCF composite. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)







Fig. 3. (a) Mode 1 bridging force vs. opening displacement for 2k carbon tufts in 4 mm thick carbon NCF composite; (b) micrograph of a partially debonded carbon tuft under load, at a displacement $2\overline{w} = 0.1 \text{ mm}$; (c) results of pull-out tests carried out on pre-delaminated single-tuft specimens manufactured as described in Section 2, but with the end loops machined off. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

2.3. Mechanical response of tufts

With reference to Fig. 3a, four stages can be identified in the crack bridging response of tufts. Initially the bridging load increases, approximately in a linear fashion, up to a critical load

in the range 50–100 N. Beyond this load, the slope of the loaddisplacement curves decreases significantly, characterising the second stage of the bridging response. Micrographs of partially loaded specimens reveal debonding cracks developing along the tufts at this stage, as shown in Fig. 3b. After reaching a peak value, P_f , the load suddenly drops. At this stage, tufts break due to tensile fibre failure, which in the majority of cases occurs in proximity of the delamination plane. However, if fibre rupture occurs at a point within the composite, frictional pull-out takes place and, after a first large drop, the load decreases gradually to zero. Due to the tufts curvature, fibres are not loaded uniformly and some fail before others, as suggested by the presence of multiple load drops during the debonding stage. This process reflects in the scatter that can be appreciated in Fig. 3a, affecting the load to failure of tufts.

The mechanical energy involved in the process of mode I delamination bridging operated by tufts, and denoted by Ψ_I , is therefore determined by the work done to deform, disbond and finally pullout the tufts. For the tested specimens, the average value of this energy is 35.6 mJ.

3. A mode I constitutive model for tufts

Micro-mechanical models of through-thickness reinforcements include those of Jain et al. [9,10] and Cox et al. [11,12], which apply both to stitches and to short rods, and those of Allegri et al. [13] and Bianchi and Zhang [5], developed specifically for *z*-pins. Structural similarities between stitches and tufts make the first two models particularly relevant for the purposes of this paper. The aim of these models is the prediction of the bridging function characteristic of a specific *z*-reinforcement. Once determined, this bridging function, or bridging law, can be represented at the meso-scale by employing discrete non-linear springs, homogenised cohesive zone models or combinations of the two [14–18].

Jain et al. [9,10] proposed a simple analytical model for a single stitch bridging a crack under mode I loading conditions. The TTR unit was assumed perfectly cylindrical and in frictional contact with the surrounding laminate. Cox et al. [11,12,19], extended this model to inclined tows, considering mixed-mode and mode II loading conditions. In these models the pre-debonding stage for the stitch is neglected. With focus on mode I loading, a frictional slip zone (l_s) is considered to start developing immediately along the stitch under tensile load. For interconnected stitches, when ls equals the embedded length of the stitch L, the load and displacement keep increasing until the thread reaches its tensile strength and breaks. In reference [11], Cox and Sridhar also accounted for the anchoring action of the horizontal thread segments of the stitch. When $l_s = L$, a further load increase is considered to be absorbed by these thread segments, which are pulled into the composite contributing to the axial displacement of the vertical thread.

Here, we present an analytical micro-mechanical model developed to capture the experimentally determined bridging mechanisms for tufts, as discussed in Section 2. The tuft is represented as a rod of cross-sectional area A, embedded in an elastic medium. It is inserted vertically and bridges a delamination at the mid-plane of a laminate. Neglecting, at this stage, the complexities of the tuft geometry and the consequences these may have on its mechanical response, the same damage mechanisms are considered to develop in both halves of the tuft. Under symmetric loading, this assumption allows to model only half of the studied domain (see Fig. 4). Unlike in the existing models [9–12], here the z-reinforcement is assumed to have an initial elastic stage, during which it is fully bonded to the composite. Where bonded, the tuft is capable of transferring load to the material it is embedded in. This load transfer causes a localised perturbation of the stresses and strains of the composite surrounding the tuft. Adopting the fundamental assumption of Cox's shear-lag theory [20], we define the shear

C. Osmiani et al./Composites: Part A xxx (2016) xxx-xxx

4



Fig. 4. Schematic of the problem described by the analytical model. Despite the irregular shape of the tuft and the fact that often failure occurs at a certain, but small, distance from the delamination plane, for simplicity only half of the system is studied, under the assumption of symmetric load and geometry.

force at the tuft interface as a linear function of the relative displacement between the tuft and the non-perturbed composite: $p_s = k_z [w^T(z) - w^C(z)]$, where k_z is a parameter defined by the elastic properties of the material around the tuft, $w^{T}(z)$ is the axial displacement of the tuft and $w^{C}(z)$ the through-thickness displacement of the undisturbed composite. As soon as a characteristic load threshold is reached, the tuft starts to debond. This process of progressive debonding is treated as a crack propagation problem, whereby a cylindrical fracture grows along the tuftcomposite interface [21-23]. Along its debonded length, the tuft stretches and slides, opposed by an assumed uniformly distributed friction force, p_0 [9,11,12,19]. This fracture mechanics-based approach to describe the debonding process is one of the main differences between the presented model and those available in the open literature, where the energy dissipated for crack propagation is generally neglected. Fully inserted tufts, as those considered in this paper, are constrained to the top and bottom surfaces of the composite they are embedded in, in the same way as stitches. Neglecting any compliance of the composite surfaces, this constraint, exerted by the horizontal thread segments on one side and the thread loops on the other, is represented in the model as an encastre support for the tuft. For simplicity, any possible interaction between neighbouring tufts has been neglected in the formulation of the model.

3.1. Governing equations

The response of the tuft has been divided into two stages called *bonded regime* and *progressive debonding regime*, respectively.

In the bonded regime, the axial equilibrium of the tuft requires

$$EA\frac{d^2w^{T}(z)}{dz^2} = k_z(w^{T}(z) - w^{C}(z)),$$
(1)

where *E* is the equivalent Young's modulus of the tuft modelled as a cylindrical rod, *A* its cross-sectional area, $w^T(z)$ is the tuft axial displacement and $w^C(z)$ the through-thickness displacement of the composite volume unaffected by the presence of the tuft. Since no forces are applied to the composite at the delamination plane, that is a free surface, the through-thickness deformation of the unperturbed composite is zero $(dw^C(z)/dz = 0)$ and Eq. (1) can be rewritten in terms of the relative displacement between the tuft and the composite, w(z), as

$$EA\frac{d^2w(z)}{dz^2} = k_z w(z).$$
⁽²⁾

Having established that w(0) = 0, since the tuft is constrained to the external surfaces of the composite by means of the surface seams and loop, and

$$EA\left(\frac{dw(z)}{dz}\right)_{z=L} = P,$$
(3)

where P is the force that needs to be applied to the tuft at the delamination plane in order for half of the system to be in equilibrium (see Fig. 4), the solution to Eq. (2) is

$$w(z) = \frac{P}{EA\alpha} \frac{\sinh(\alpha z)}{\cosh(\alpha L)},$$
(4)

where $\alpha = [k_z/(EA)]^{1/2}$. Eq. (4) defines the axial displacement of the tuft as a function of the axial coordinate *z* and the external load *P* (see Fig. 4). At the delamination plane (*z* = *L*), this displacement becomes

$$\overline{w} = w(z = L) = \frac{P}{EA\alpha} \tanh(\alpha L), \tag{5}$$

which corresponds to half of the relative displacement between the surfaces of a bridged crack.

If, on the other hand, we assume the tuft to be debonded over a generic length l_d , the axial equilibrium of the tuft is defined by the following system of equations:

$$EA\frac{d^2w_b(z)}{dz^2} = k_z w_b(z) \qquad \text{if } 0 \leqslant z \leqslant L - l_d, \tag{6a}$$

$$EA\frac{d^2w_d(z)}{dz^2} = p_0 \qquad \text{if } L - l_d \leqslant z \leqslant L, \tag{6b}$$

where the subscripts b and d refer to the bonded and debonded portions of the tuft, respectively. Eq. (6a) must satisfy the geometric and natural boundary conditions defined as

$$w_b(0) = 0, \tag{7a}$$

$$EA\left(\frac{dw_b(z)}{dz}\right)_{z=L-l_d} = \widehat{P},\tag{7b}$$

with $\hat{P} = P - p_0 l_d$, following from simple equilibrium considerations. The tuft axial displacement in the domain $z = [0, L - l_d]$, is

$$w_b(z) = \frac{P - p_0 l_d}{EA\alpha \cosh(\alpha L - \alpha l_d)} \sinh(\alpha z).$$
(8)

The solution to Eq. (6b) can now be obtained by imposing:

$$w_d(L-l_d) = \hat{w},\tag{9a}$$

$$EA\left(\frac{dw_d(z)}{dz}\right)_{z=L} = P,\tag{9b}$$

where \hat{w} follows from Eq. (8) for $z = L - l_d$. Satisfaction of the boundary conditions implies

$$w_{d}(z) = \frac{1}{EA} \left[\frac{p_{0}z^{2}}{2} + (P - p_{0}L)z + \frac{P - p_{0}l_{d}}{\alpha} \tanh(\alpha L - \alpha l_{d}) + \frac{p_{0}}{2} \left(L^{2} - l_{d}^{2} \right) - P(L - l_{d}) \right],$$
(10)

which defines the *z*-displacement of the tuft for $L - I_d \le z \le L$, in the regime of progressive debonding. At the delamination plane, this displacement is

$$\bar{w} = w_d(L) = \frac{P}{EA} \left[l_d + \frac{\tanh(\alpha L - \alpha l_d)}{\alpha} \right] - \frac{p_0 l_d}{EA} \left[\frac{l_d}{2} + \frac{\tanh(\alpha L - \alpha l_d)}{\alpha} \right].$$
(11)

It can be noted that for a zero debonding length ($l_d = 0$), Eq. (11) correctly coincides with Eq. (5). On the other hand, in the case of a fully debonded tuft, the displacement at the delamination plane becomes

$$\overline{w}_{l_d=L} = \frac{L}{EA} \left[P - \frac{p_0 L}{2} \right],\tag{12}$$

which is independent from the elastic foundation stiffness, k_z .

3.2. Debonding criterion

When the process of debonding starts, a crack is considered to propagate along the lateral surface of the cylindrical tuft. The energy required for an infinitesimal increment of the debonded length is $G^{C,int}l_pdl_d$, where $G^{C,int}$ is referred to as 'fracture toughness of interface' and l_p is the cross-section perimeter of the *z*reinforcement. Since friction is assumed to act along the debonded length of the tuft, for the principle of stationarity of the total energy, the energy available for fracture growth is

$$G^{int}l_p dl_d = dW_{ext} - dU_{fr} - dU_e,$$
⁽¹³⁾

where G^{int} is the equivalent mode II strain energy release rate of the interface, W_{ext} is the total work done by conservative forces on the system, U_{fr} is the energy dissipated by friction and U_e is the elastic strain energy. It follows that the energy available for a unit increment of the crack surface is

$$G^{int} = \frac{d(W_{ext} - U_{fr} - U_e)}{l_p dl_d}.$$
 (14)

The numerator of Eq. (14) is the energy difference between a state in which the debonded length is l_d and a second state in which the debonded length is $l_d + dl_d$, as illustrated in Fig. 5. Following this criterion, and with few manipulations (see Appendix A), Eq. (14) can be re-written as

$$G^{int} = \frac{1}{l_p} \left[P \frac{d\overline{w}}{dl_d} + k_z w_b^2 (L - l_d) - p_0 w_d (L - l_d) \right].$$
(15)

Fig. 5. Schematic of the two states of the system considered for the definition of the strain energy release rate of interface. From state 1 to state 2 the only variation consists in an increase Δl_d of the debonded length of the tuft.

By substituting the expressions of $\frac{dW}{dl_d}$, $w_b(L - l_d)$ and $w_d(L - l_d)$ in Eq. (15), one obtains

$$G^{int} = \frac{\tanh(\alpha L - \alpha l_d)}{l_p E A \alpha} \left[\frac{\alpha \tanh(\alpha L - \alpha l_d)}{2} \left[3P^2 - 4P p_0 l_d + p_0^2 l_d^2 \right] + p_0^2 l_d - 2P p_0 \right].$$
(16)

For $l_d = 0$ and $G^{int} = G^{C,int}$, Eq. (16) provides the critical load marking the onset of debonding

$$P_{C} = \frac{p_{0} \pm \sqrt{p_{0}^{2} + \frac{3}{2}G^{C,int}l_{p}EA\alpha^{2}}}{\frac{3}{2}\alpha\tanh(\alpha L)}.$$
(17)

The critical load depends on both the fracture toughness of interface and the friction force acting along the debonded portion of tuft. To initiate debonding, the applied load needs to be high enough to provide sufficient energy for the crack to propagate and to equilibrate the forces, including p_0 , acting along the TTR.

During debonding, $G^{int} = G^{C,int}$ and $P = P_C + \Delta P$. The substitution of $G^{C,int}$ in place of G^{int} in Eq. (16) provides a relation between load and debonded length during the regime of progressive debonding. For a given l_d , the corresponding load value is determined:

$$P = \frac{p_0[l_d \alpha \tanh(\alpha L - \alpha l_d) + 1] \pm \frac{\sqrt{D}}{2}}{\frac{3}{2}\alpha \tanh(\alpha L - \alpha l_d)},$$
(18)

where

$$D = p_0^2 \{ l_d \alpha \tanh(\alpha L - \alpha l_d) [\alpha \tanh(\alpha L - \alpha l_d) l_d + 1] + 4 \}$$

+ $6G^{C,int} l_p EA\alpha^2.$ (19)

4. Identification of model parameters

The proposed model requires the evaluation of a series of geometrical and physical input parameters. The cross-sectional area and perimeter of the tuft, *A* and l_p , and the composite thickness, 2L, can be determined from micrographs. The equivalent Young's modulus, *E*, the friction force, p_0 , the foundation stiffness, k_z , and the equivalent mode II fracture toughness of interface, $G^{C,int}$, are physical parameters requiring ad - hoc experiments to be determined.

4.1. Tufting thread characterisation

The most common architecture of commercially available continuous carbon fibre threads consists of a given number of fibre bundles, called yarns, interlaced together so that each yarn follows a helical path. The structure of the tufting thread resembles that of a rope, without a central core. When the thread is embedded into a preform, the preform architecture controls the final shape of its cross-section and the length of its perimeter, l_p . For a specific preform, l_p and A need to be measured from micrographs of tufted samples. As for the parameter defining the equivalent elastic modulus of the tuft in the model, *E*, this is determined by the elastic properties and volume fractions of the fibres and matrix constituting the tuft, as well as by the *in situ* geometry of the tuft. The latter is influenced both by the in-plane fibre architecture and stacking sequence of the preform and by the process of consolidation. This makes the identification of the final shape of the tuft, and consequently of the equivalent elastic modulus E, not trivial, which is the reason why E has been selected as one of the calibration parameters of the model.

5

6

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C. Osmiani et al./Composites: Part A xxx (2016) xxx-xxx

4.2. Elastic foundation stiffness

As explained in Section 3, stress transfer between the tuft and the composite over the bonded length of the tuft has been taken into account by modelling the material around the tuft as a bed of tangential springs of stiffness k_z [20]. A way of approximately quantifying this stiffness parameter is by means of Finite Element (FE) calculations. The model illustrated in Fig. 6 has been used to characterise k_z as a function of the tuft equivalent elastic modulus E and the tuft embedded length, L. In this model, the tuft is described as a cylindrical rod of cross sectional area A (defined in Section 3.1), fully bonded to the surrounding medium. The FE analyses have been carried out using Abaqus 6.14. A static-general step has been implemented using C3D8 elements. A linearly varying displacement field has been applied at the nodes of the tuft, as shown in Fig. 6, and the reaction force on the top surface of the composite has been requested in output. Elastic transversely isotropic properties have been assigned to both the tuft and the composite.

Fig. 7a shows that k_z has a strong dependency on the thickness of the composite, which in the analysed models corresponds to the embedded length of the tuft, L. This result would suggest that the adoption of a constant k_z value throughout the process of progressive debonding of the tuft is not valid, and it can only be accepted as a consequence of the low sensitivity of the model to k_{τ} in that stage of the bridging response, as demonstrated by the plot in Fig. 7b. k_z varies not only with the embedded length of the tuft, but also with the boundary conditions of the tufted unit-cell. This means that its value would need to be re-evaluated for every new tufted structure under investigation. However, Fig. 7b shows that any variation of k_{z} has little influence on the constitutive law of the tufted interface, within the limits of experimental repeatability. This implies that the curve calibrated on the results of the single-tuft tests has general validity, i.e. it can be used to predict the delamination behaviour of any tufted composite structure manufactured with the same materials and fibre architecture of the single-tuft specimens, independent of any variation of the boundary conditions. The effect of a different composite thickness can be predicted with the use of the analytical model only. Fig. 7c further shows that k_7 is also a function of the equivalent longitudinal modulus of the tuft, E; hence, in order to be able and determine the value of k_z for a particular system, it is necessary to know the values of L and E for that specific system. However, since E is a calibration parameter, its value cannot be known a priori. This obstacle can be overcome using the empirical formula below, which

Fig. 6. Finite element model of a delaminated single-tuft composite panel with a $[(0/90)_s]_2$ layup. Exploiting symmetry, only one-eighth of the system has been modelled. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

establishes a relation between *E* and k_z , based on the results of Fig. 7d:

$$\frac{P}{\overline{w}} = aE + b = \frac{EA\sqrt{\frac{k_z}{EA}}}{\tanh\left(\sqrt{\frac{k_z}{EA}}L\right)}.$$
(20)

For the system described in Section 2, the coefficients *a* and *b* have been determined to have values $0.1571(\pm 0.0196)$ mm and 6976 (± 1528) N/mm, respectively. Eq. (20) has been used during the process of model calibration to determine which value of k_z corresponded to the *E* value providing the best fit.

4.3. Friction force

The friction force, p_0 , acting along the debonded length of the tuft, is influenced by two different mechanisms. First, thermal residual stresses due to post-cure cool down affect the interfacial friction between the tuft and the composite. Secondly, the irregularity of the tuft longitudinal profile promotes mechanical interlocking between the tuft and the surrounding resin material. Since these irregularities have not been accounted for in the proposed model formulation, their effect reflects on the value of p_0 , as well as on the values of the other material parameters of the model. The quantification of p_0 , which in this study is assumed to be uniformly distributed [9,12,11], has been carried out by means of pull-out tests on single-tuft specimens in which the tuft loop has been machined off. The load-displacement curves obtained for the specimens described in Section 2 are reported in Fig. 3c. Given the average maximum force P_{PO} in the tuft at pullout initiation, the tuft cross-section perimeter l_p , and the effective pull-out length, L_{PO} , the equivalent uniform friction stress opposing the sliding of the tuft can be calculated as [6]

$$\tau_0 = \frac{P_{PO}}{l_p L_{PO}} \tag{21}$$

For the considered composite system, $\tau_0 = 36$ (COV = 16.7%) MPa and the equivalent friction force distributed along the tuft, defined as $p_0 = \tau_0 l_p$, is 80.6 N/mm.

4.4. Fracture toughness of interface

The process of debonding has been modelled as the propagation of a crack at the interface between the tuft and the composite, in pure mode II. This is a simplification derived from the assumption of the tuft as a straight rod. In reality, tufts often display curved profiles with irregular perimeters, which translates into a variable stress-state at the tuft-composite interface and a debonding crack propagating in mixed-mode. The mode mixity varies during crack growth, as therefore does $G^{C,int}$. Depending on the *in situ* shape of the tuft, debonding may affect only a portion of its lateral surface, i.e the portion subjected to the highest tensile and shear stresses. The assumption of a debonding crack propagating uniformly and symmetrically in the two halves of the delaminated tufted composite is expected to reflect inevitably on the input value of $G^{C,int}$. which is therefore difficult to be determined a priori. Based on these considerations, the fracture toughness of interface has been selected, together with E, to be calibrated on experimental data sets obtained from single-tuft tests.

5. Calibration of the model

For calibration purposes, the proposed analytical model has been implemented in MATLAB and formulated in two steps, the first representing the *bonded regime* and the second describing pro-

C. Osmiani et al. / Composites: Part A xxx (2016) xxx-xxx

Fig. 7. (a) Variation of k_z with the embedded length of the tuft, from the model of Fig. 6; (b) sensitivity of the analytical model to k_z ; (c) variation of k_z with the tuft Young's modulus; (d) plot of P/\overline{w} vs. *E* for three thicknesses of the composite panel, fixed tuft geometry and fixed composite mechanical properties.

gressive debonding. The critical load, P_c , and corresponding critical displacement, \overline{w}_c , are determined by means of Eqs. (17) and (5). The bridging law in the *bonded regime* is obtained by discretising the displacement range up to $\overline{w} = \overline{w}_c$, and evaluating the corresponding values of *P* through Eq. (5). The load-displacement curve is linear up to $P = P_c$. The length of debonding, I_d , is given as input to the second part of the script. Given the values of I_d , *P* can be determined from Eq. (18) and the corresponding displacement obtained by means of Eq. (11). In this regime, the bridging law is non-linear due to the energy dissipated by the system for increasing the debonded length of the tuft and for overcoming friction. Table 1 provides a summary of the model input parameters, with their experimentally determined values. The only two parameters requiring calibration are the equivalent axial elastic modulus of the tuft, *E*, and the fracture toughness of the tuft-composite interface,

Table 1

Analytical model: input parameters and results of calibration process (COV in brackets).

Symbol	Measured	From calibration
L	2 mm	-
Α	0.27	-
l_p	2.24 (6.3%) mm	-
\dot{p}_0	80.6 N/mm (see Section 4.3)	-
ϵ_{f}	0.05 (20%) mm	-
Ψ_I	35.6 mJ ^a	-
$G_{I}^{C,tuft}$	131.8 kJ/m ²	-
E	_	14.64 GPa
k₂	_	2.176E+4 N/mm ²
$\tilde{G}^{C,int}$	-	0.543 kJ/m ²
$G_I^{C,tuft}$	-	130.4 kJ/m ²

^a Obtained neglecting the contribution of partial pull-out events.

 $G^{C,int}$, as discussed in Section 4. These unknown parameters have been identified by means of the Genetic Algorithm (GA) routine in MATLAB [13]. The cost function to be minimised in the GA optimisation scheme is

$$C = \sqrt{\epsilon_{P\overline{w}}^2 + \epsilon_{\Psi_b}^2 + \epsilon_{\Psi_d b}^2},\tag{22}$$

where $\epsilon_{P\overline{W}}^2$ is the relative mean square error associated with the averaged load-displacement data points, and $\epsilon^2_{\Psi_b}$ and $\epsilon^2_{\Psi_{ab}}$ are the relative mean square errors of the work done by the external forces in the bonded and progressive debonding regimes (neglecting the work of partial frictional pull-out, if present), respectively. The optimisation has been carried out considering 50 combinations of populations of 50 individuals each. An initial range of values has been assigned to each calibration parameter: [0.1240,000] MPa and [02] kJ/m² for *E* and *G*^{*C*,*int*}, respectively. The upper bound for the tuft equivalent Young's modulus has been chosen as the tensile modulus of the impregnated carbon fibre yarns forming the tuft (Tenax[®]-I HTA40 H15 1K 67tex [24]), whereas G^{C,int} is allowed to vary in a range of carbon-epoxy laminates common values of fracture toughness [25]. The values of the calibrated parameters are reported in Table 1 and the resulting calibrated bridging law is displayed in Fig. 8. As mentioned in Section 2.3, the failure load of the tuft is a function of the tuft *in situ* topology and cannot therefore be determined from independent tensile tests on impregnated tufting threads. Hence, in this paper, an experimentally-based failure criterion deduced from mode I testing of single-tuft coupons has been selected. In particular, the calibrated bridging law of Fig. 8 has been interrupted at the average measured deformation at failure of the tested carbon-fibre tufts (see Table 1). The debonded length corresponding to this deformation is $l_s = 1.77$ mm, 88.5% of the embed-

7

Fig. 8. Result of the calibration process for the input parameters listed in Table 1. The continuous curves in the plot are the maxima and minima envelopes of the experimental data presented in Fig. 3a. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

ded length of the tuft. As for the energy involved in the process, the current model predicts that the energy loss due to friction and to debonding contribute to approximately 40% and 13% of the total energy of the system, respectively. Thus, although the energy dissipated by friction is dominant, the energy spent for crack propagation at the tuft-composite interface is not negligible. This supports the modelling strategy detailed in Section 3.

The determined E and $G^{C,int}$ parameters can now be used to predict the response to delamination of tufted meso-scale structures with the same characteristics, in terms of tufting thread material and preform material and architecture, of the single-tuft specimens the model has been calibrated on.

6. Validation of the model

6.1. Mode I double cantilever beam test on tufted specimens

The new micro-mechanical analytical model has been validated by comparing meso-scale FE simulations to delamination tests of DCB specimens. Fig. 9 shows the schematics of sample dimensions and tuft row positioning used in the DCB tests. The longitudinal axis of the DCB specimen is parallel to the 0° ply orientation at the mid-plane of the specimen. The tufts were inserted normal to the laminate plane, in a square pattern with 5.6 mm × 5.6 mm spacing, resulting in an areal density of tufts of 0.86%. The tufting seams were aligned orthogonal to the 0° ply orientation, with the first tuft row positioned 15 mm from the end of the release film, covering a total delamination length of 100 mm. A 10 µm thick PTFE film, at the mid-plane of the preform, extends 61 (±0.5) mm from the edge producing an initial crack length of 50 mm in the consolidated panels. Each panel was injected using the VARTM process, resulting in a consistent laminate thickness of 4.0 (± 0.01) mm and a fibre volume fraction of 56%.

Delamination tests on the DCB specimens were performed on a universal test machine Instron 5500R, with 5 kN load cell, at a constant crosshead displacement rate of 1 mm/min. In absence of standardised procedures for testing through-the-thickness reinforced bi-directional laminates, mode I tests were performed following the guidelines developed for 'unreinforced' unidirectional composites [26]. Prior to testing, the initial pre-crack was propagated for 5 mm from the film insert. The new crack tip ensured the measured initiation values to be independent of the insert film thickness. Finally, for monitoring purposes, one side of each beam was coated with a thin layer of brittle white spray-on paint and marked with millimetre increments to enable visual monitoring of the crack tip position. On the opposite side, a black and white speckle pattern was applied for monitoring of crack propagation and for measurement of opening displacements with a Digital Image Correlation (DIC) system (Limess VIC 2D and VIC 3D).

To avoid premature flexural failure of the 2 mm thick beams of the tufted DCB specimens (see Fig. 10), 3 mm thick strips of cured unidirectional composite were bonded to both sides of the samples, as suggested by other authors [27,28]. The unidirectional laminate had 10 layers of carbon fibre uni-weave fabric (OCV Technical FabricsTM, 12k Grafil 34-700) with an areal weight of 310 g/m^2 , injected with MVR444 epoxy resin.

6.2. Meso-scale FE model

6.2.1. Modelling tufted interfaces at the meso-scale

Cohesive zone modelling is the approach selected to model the delamination behaviour of the tufted DCB specimens of Section 6.1. The application of this approach implies the use of cohesive elements to simulate the delamination behaviour of both the unreinforced interface and the discrete through-thickness reinforcements [5]. The former is governed by a bilinear traction-separation law in which the onset of damage is related to the interfacial strength, i.e., the maximum traction on the traction-separation jump relation, and the propagation of delamination follows a fracture mechanics criterion. When the area under the traction-separation jump relation is equal to the intrinsic fracture toughness of the resin, the tractions are reduced to zero and a new delamination front is formed [29,30]. As for the cohesive elements used to simulate the behaviour of the tufted regions, their constitutive law has been obtained using the micro-mechanical model presented in this paper (see Fig. 11). This constitutive law describes the initial localised elastic deformation of the tufts and material surrounding them (bonded regime), followed by progressive debonding of the tufts up to failure. It should be noted that the localised deformation taking place in the material around the bonded tufts can only be accounted for via the constitutive law of the interface elements, as the meshes of meso-scale models are generally too coarse to capture it. To avoid any further unrealistic through-thickness deformation of the material above and below the bridged interface

Fig. 9. Schematic of specimen configuration for mode I DCB delamination testing. The same fabric, resin and tufting thread were used as in the corresponding single-tuft tests. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

Fig. 10. (a) DIC image of untabbed DCB specimen, failed in flexure at first tuft row; (b) large scale tuft bridging in DCB sample with 0.86% tufting, reinforced by UD stiffeners. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

elements, columns of rigid elements have been inserted in the arms of the DCB model at the tuft locations. The implementation of the mode I constitutive law of tufts into this finite element framework has been achieved via a user-defined constitutive formulation in which an arbitrary normalised traction-separation law can be prescribed.

6.2.2. Model description

The tufted DCB specimens of Section 6.1 have been simulated in the explicit finite element code, LS-DYNA v971-R7.1. 8-node selectively reduced-integration solid elements with hourglass control were employed to simulate both the composite NCF and UD beams with at least 6 elements in the through-thickness direction. This was done for both sub-laminates in order to capture the bending stiffness and laminate rotations. Symmetry conditions were exploited by modelling only one half of the specimen and taking into account the periodic arrangement of the tufts. In the tufted beam, tufts were modelled discretely with rigid elements and a tuft-to-tuft distance of 5.6 mm (see Fig. 12). The displacement rate was approximately 1 mm/s (defined initially by a smooth ramp rate followed by a constant velocity boundary condition imposed at the load application points), which made dynamic effects negligible. Mass scaling was used to reduce the solution time. A summary of the elastic homogenised properties assigned to the composite beams and a list of the cohesive parameters used to describe the mechanical responses of the un-tufted and tufted interfacial regions are provided in Tables 2 and 3, respectively. Due to the presence of the stiffening UD tabs, the average initiation fracture toughness measured for the tested DCB specimens has been used as the mode I matrix fracture toughness and its value is $G_I^{C,resin} = 0.257 \text{ kJ/m}^2$ (COV = 5.9%). The fracture toughness of the tufted interface, $G_t^{C,tuft}$, has been calculated as the overall bridging work of the tuft, Ψ_l , divided by its cross-sectional area A. Following a mesh convergence study, in which the results were checked during the elastic and crack propagation phases, a minimum cohesive element length of 0.25 mm in the direction of crack propagation was adopted. This allowed the model to have three cohesive elements in the fully developed process zone ahead of the crack tip, in agreement with current meshing guidelines for cohesive zone models [31,30].

6.3. Results and discussion

Fig. 13 shows a comparison of the predicted and experimental DCB test results, for a tufted areal density of 0.86%. The numerical curve has been obtained by implementing the calibrated analytical traction-separation law in the constitutive cohesive zone model of the tuft ($G_l^{C,tuft}$ = 130.4 kJ/m²). It is clear that the delamination initially propagates through the untufted region, up to the first tuft row. Here, the localised bridging of the tufts causes a significant load increase, i.e. almost 130%. After failure of the first row of tufts, a large-scale bridging zone fully develops. For the laminate and DCB configuration considered here, the bridging region comprises two rows of intact tufts. With further opening displacement, the crack propagates unstably and is arrested by the presence of the localised reinforcements, causing the rows of tufts to progressively stretch and fail [15]. In this region of the load-displacement curve,

Fig. 11. Traction-separation laws of localised tuft and of pure resin employed in cohesive elements. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

C. Osmiani et al./Composites: Part A xxx (2016) xxx-xxx

Fig. 12. LS-DYNA finite element model of mode I DCB specimens. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

Table 2				
Elastic homogenised properties of NCF and UD carbon composites and MVR444 resin.				
T.		E (CD)	E (CD)	

Туре	E_1 [GPa]	E_2 [GPa]	E_3 [GPa]	G ₁₂ [GPa]	G ₁₃ [GPa]	G ₂₃ [GPa]	v ₁₂ [-]	v ₁₃ [-]	v ₂₃ [-]
NCF UD	70.3 107	70.3 107	12.1 11	4.1 4.0	4.05 4.0	4.05 4.07	0.042 0.271	0.344 0.271	0.344 0.344
Туре	E[GPa]	v[-]							
MVR444	3.1 ^a	0.35 ^a							

^a Neat resin properties stated by the manufacturer.

Table 3

Parameters of cohesive laws.

Туре	σ_{I}^{0} [MPa]	$G_{IC} [kJ/m^2]$
Resin interface	60	0.257
Tuft (analytical)	1020	130.4

Fig. 13. Comparison between experiments and simulation in terms of applied force vs. opening displacement (tufted laminate: 0.86% tufted areal density). (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

the FE model underestimates the bridging response of the tufts, showing a more pronounced *saw-tooth* behaviour compared to the experimental curves. This could be corrected by considering the effect of fibre bridging, caused by the in-plane fibres of the fabric, on crack propagation (see Fig. 9).

7. Conclusions

A multi-scale modelling framework for predicting the mode I delamination behaviour of tufted composites has been presented and validated. It applies to the case of structures tufted through their entire thickness, with loops long enough to anchor the tufts and prevent them from pulling-out under load. The proposed multi-scale approach is based on a micro-mechanical model describing the mode I response of bridged interfaces, coupled with a meso-scale cohesive zone formulation for tufted structures. In the paper, this strategy has been applied successfully for the prediction of the delamination behaviour of 4 mm thick double cantilever beam specimens containing an array of tufts at 0.86% areal density.

The modelling approach presented has the advantage of requiring the calibration of only two parameters at the micro-scale (i.e. at single-tuft level), namely the equivalent Young's modulus of the cylindrical tuft and the equivalent mode II fracture toughness of the tuft/composite interface, and it captures the main bridging mechanisms observed for tufts under tensile loading, i.e. initial linear elastic deformation of the bonded tuft, followed by a progressive debonding event and frictional sliding of the debonded tuft, which are responsible for the non-linearity of its response. Despite the level of idealisation, the micro-scale model presented, when implemented into interface elements at the meso-scale, was capable of predicting the stiffness and first load peak of the tested DCB specimens within the limits of experimental repeatability.

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Appendix A. On the formulation of the debonding process

Eq. (16) has been obtained by imposing the limit for $\Delta l_d \rightarrow 0$ to the energy variation characterising the system described in Fig. 4, when the length of the debonding crack increases of Δl_d . With reference to Fig. 5, in state 1 the energy of the system is

$$E_{1} = \int_{0}^{\overline{w}_{1}} P(w)dw - \frac{1}{2} \int_{0}^{L-l_{d}} k_{z}w_{b}^{2}(s)ds - p_{0} \int_{L-l_{d}}^{L} \int_{0}^{w_{d}(s)} dwds - \frac{1}{2} \int_{0}^{L-l_{d}} \frac{N_{b}(s)^{2}}{EA} ds - \frac{1}{2} \int_{L-l_{d}}^{L} \frac{N_{d}(s)^{2}}{EA} ds$$
(A.1)

and, similarly, in state 2 the energy is

$$E_{2} = \int_{0}^{\overline{w}_{2}} P(w)dw - \frac{1}{2} \int_{0}^{L-(l_{d}+\Delta l_{d})} k_{z}w_{b}^{2}(s)ds - p_{0} \int_{L-(l_{d}+\Delta l_{d})}^{L} \int_{0}^{w_{d}(s)} dwds - \frac{1}{2} \int_{0}^{L-(l_{d}+\Delta l_{d})} \frac{N_{b}(s)^{2}}{EA} ds - \frac{1}{2} \int_{L-(l_{d}+\Delta l_{d})}^{L} \frac{N_{d}(s)^{2}}{EA} ds$$
(A.2)

where

$$\overline{w_2} = \overline{w_1} + \frac{\Delta \overline{w}_1}{\Delta l_d} \Delta l_d. \tag{A.3}$$

The energy variation between state 1 and state 2 becomes:

$$E_{2} - E_{1} = \int_{\overline{w}_{1}}^{\overline{w}_{1} + \frac{\Delta w_{1}}{\Delta d_{d}} \Delta l_{d}} P(w) dw + \frac{1}{2} \int_{L-(l_{d} + \Delta l_{d})}^{L-l_{d}} k_{z} w_{b}^{2}(s) ds$$

$$- p_{0} \int_{L-(l_{d} + \Delta l_{d})}^{L-l_{d}} w_{d}(s) ds$$

$$- \frac{1}{2} \int_{l-(l_{d} + \Delta l_{d})}^{L-l_{d}} \frac{\left(N_{b}(s)^{2} - N_{d}(s)^{2}\right)}{EA} ds.$$
(A.4)

Finally, for Δl_d small, Eq. (A.4) can be re-written in the form

$$E_{2} - E_{1} = P(\overline{w}_{1}) \frac{d\overline{w}_{1}}{dl_{d}} dl_{d} + \frac{dl_{d}}{2} k_{z} w_{b}^{2} (L - l_{d}) - p_{0} w_{d} (L - l_{d}) dl_{d}$$
$$- \frac{dl_{d}}{2} \frac{\left(N_{b} (L - l_{d})^{2} - N_{d} (L - l_{d})^{2}\right)}{EA}$$
(A.5)

in which $N_b(L - l_d) = N_d(L - l_d)$ for continuity.

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