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Multi area load frequency control using particle swarm optimization and fuzzy rules

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Abstract

This paper present heuristics based study of multi area power network. Heuristic procedures involving Particle Swarm Intelligence and Fuzzy based inferences have been employed to effectively obtain the optimized gains of PID controller. Any change in the load demand causes generator's shaft speed lower than the pre-set value and the system frequency deviates from the standard value results in malfunctioning of frequency relays. A five area load frequency model is constructed in Matlab/simulink by implementing the PID (Proportional, Integral and Differential) controllers to control the frequency deviations. The effect of interconnection of multi area power system as ring connection has been discussed. Simulations performed show the effectiveness of the current approach over simple fuzzy inferences in terms of performance as well as execution efficiency.

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Keywords: Load Frequency Control; Particle Swarm Optimization ; Fuzzy Rules; Error Analysis.

1. Introduction

The term LFC(Load Frequency Control) in an interconnected power system which consists of large number of control areas, signifies to maintain the frequency of each area within predefined standard limits and to keep tie-line power flows within some pre-specified tolerances by adjusting the MW (mega-watt) outputs of the generators so as to accommodate fluctuating load demands. A well designed and controlled power system provides damping to the transients produced due to the load and system disturbances.

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Nomenclature			
df_i	Frequency deviation	$dP_{mech_n}^i$	Governor Valve position
$dP_{c\ i}$	Controller output actuation signal	dP_{tie-N}	Net tie-line power flow
dPL_i	Change in demand power or load	H_i	Equivalent inertia constant
D_i	Equivalent damping coefficient		
T_{ij}	Tie-line synchronizing coefficient with area j		
$G_i(s)$	Transfer function Turbine-Generator unit in cascade.		
B_i	Frequency bias	\check{v}	Area interface
R_i	Droop characteristic	AFCE	Area frequency control error
$\phi_{n\ i}$	Participation factors	$\dot{M}(s)$	Governor-turbine dynamic model
$K_{PID}(s)$	Dynamic PID(Proportional Integral and Differential) controller		

Also, it should provide acceptable high level of power quality while maintaining both voltage and frequency within desirable tolerance limits. Load-frequency control (LFC), or automatic generation control, is a very important issue in power system operation and control for supplying sufficient and both good quality and reliable electric power [1-3]. In order to improve the transient response, advanced control techniques have been proposed, which include PID controller incorporated with soft computing and optimization techniques such as Fuzzy logic, PSO (Particle Swarm Optimization) Genetic Algorithms and advanced self adjusting fuzzy[4-13] and [14,15]. GA and PSO tuned fuzzy controller for AGC in three area power system proposed in [16-19]. Nowadays the complexity issues in power system are being solved with the use of GAs, PSOs, bacterial foraging optimization algorithm(BFA). But in the present scenario due to penetration of the DFIG-WT(Doubly fed Induction Generator based Wind Turbine) and diesel based power system into the main grids, give rise to stability issues of system frequency, active and reactive power under load demand disturbances[20-23]. So, in keeping view of the present status of the power system, a DFIG-WT and diesel based power system is added in the LFC model of the three area power system which is consist of thermal-hydro. In this paper a five control area LFC model has been made using Matlab/simulink. The performance of fuzzy and PSO based PID controllers is verified for the same load disturbances. An error based comparative analysis has been carried out to reach effective conclusion as described at the last.

2. Control area model description

2.1. Load Frequency Model of a single area with non-reheat steam and Hydro Generating Unit.

The LFC model of control area ' i ' consist of conventional steam unit connected to a hydro unit in a power system comprised of N control areas interconnected through same tie-line is represented by the block diagram [1] as shown in fig.1. The output of power system block is change in system frequency (df_i) corresponds to load disturbances and act as feedback signal which is compared with reference signal at the input and give rise to an error signal AFCE (Area Frequency Control Error) [1-3]. The regulated output of the controller is given to governor valve to reduce down the deviations in the system frequency, tie line power and mechanical power. For N interconnected control areas, the total change in tie-line power between area i and other areas is given by [1] as follows:

$$dP_{tie,i} = \sum_{\substack{j=1 \\ j \neq i}}^N dP_{tie,ij} = \frac{2\pi}{s} \left[\sum_{\substack{j=1 \\ j \neq i}}^N T_{ij} df_i - \sum_{\substack{j=1 \\ j \neq i}}^N T_{ij} df_j \right] \quad (1)$$

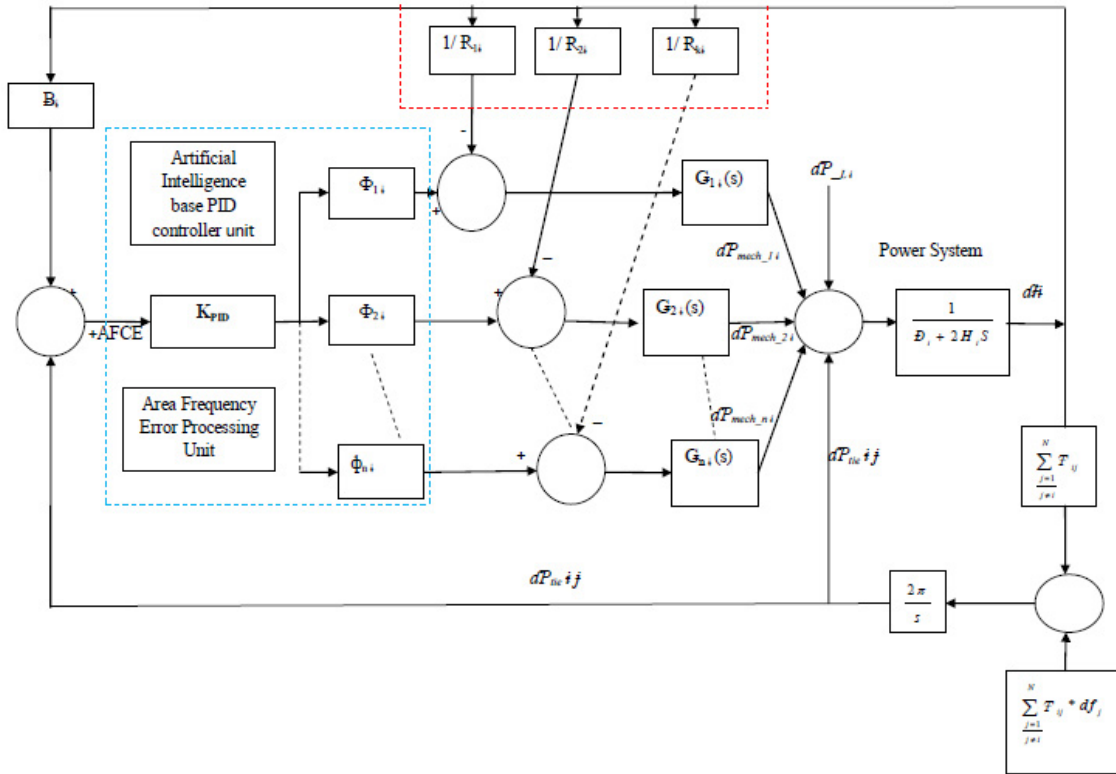


Fig.1. Control area i with Primary and Artificial Intelligence based Control unit [1]

The output of PID controller i.e. u_i proportional to area control error and is given by [1] as:

$$u_i = dPci = k_{pi} AFCE_i + k_{ii} \int AFCE_i + k_{Di} \frac{d}{dt} AFCE_i \tag{2}$$

Where K_{pi} , K_{ii} and K_{Di} are the gain constants of PID. $dPci$ is the regulated actuating control signal. A control area is known as DISC-CA and a generating unit as GENC-GU [1]. GC is the gencos contribution matrix which shows the contribution of each GU in an N control area system. The rows of a GC correspond to GU and columns to corresponds CA . For example, for a large scale power system with m control areas (CA) and k generating units (GU), the GC will have the following structure [1].

$$GC = \begin{bmatrix} cf_{11} & cf_{12} & cf_{1(m-1)} & cf_{1m} \\ \vdots & \vdots & \vdots & \vdots \\ \vdots & \vdots & \vdots & \vdots \\ cf_{n1} & cf_{n2} & cf_{k(m-1)} & cf_{km} \end{bmatrix} \tag{3}$$

Any entry in a GC that corresponds to a contracted load by a CA, demanded from the corresponding GU, must be reflected to the control area system. The generation of each GU must track the contracted demands of CA in steady state. The desired total power generation of GU_i in terms of GC entries can be calculated as[1]

$$dP_{mi} = \sum_{j=1}^N cf_{ij} dP_{Lj} \tag{4}$$

2.2. Dynamic Modelling of DFIG-Wind Turbine and Diesel Generator

2.2.1 DFIG-Wind Turbine based Generating Units GU:

DFIG (Doubly Fed Induction Generator)-based wind turbine is represented by the small-perturbation model shown in Fig.2, [24]. T_r is the transducer time constant, T_w is the washout filter time constant for DFIG area dP_{N_ref} is the Incremental Wind Turbine active Power reference (non-conventional generation) and ω is the wind Turbine speed. The system behaviour depends on the choice of network parameters and DFIG-based wind turbine speed controllers are K_{wp} and K_{wi} . If several generators are connected to the system, the equivalent regulation droop can be determined from the eq. (5) as follows [24]:

$$\frac{1}{R} = \frac{1}{R_1} + \frac{1}{R_2} + \dots + \frac{1}{R_n} \tag{5}$$

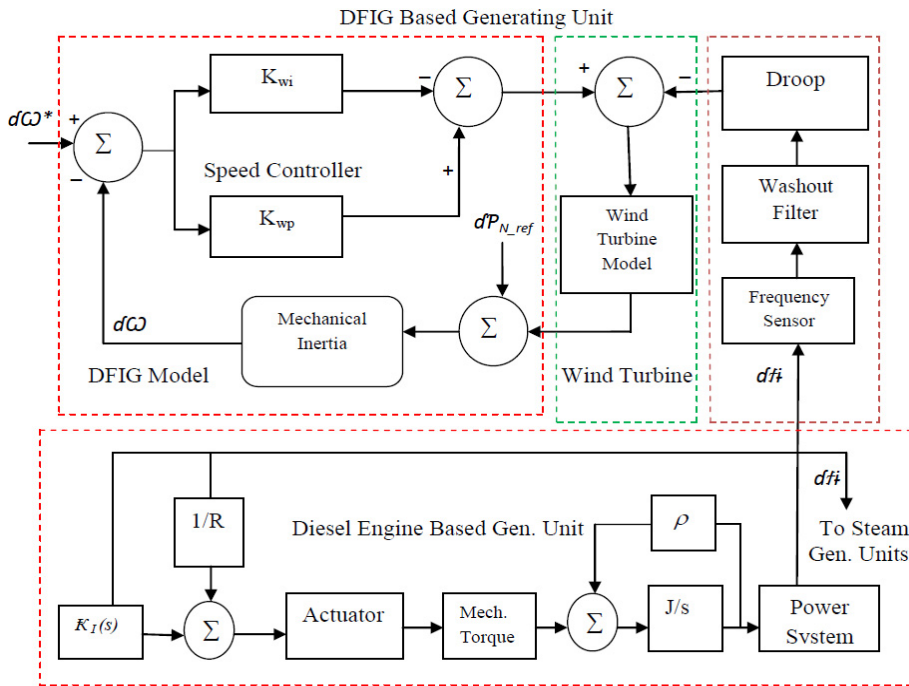


Fig.2. DFIG and Diesel based Area Load frequency control [24]

2.2.2 Diesel -Generator based Generating Unit GU

There are many methods already proposed for modelling a diesel generator [24,25]. It consists of fuel actuator system to regulate the speed of the engine and is usually represented as a first order network, which is characterized by gain K_2 and time constant t_2 . Fig.2. shows the actuator block having transfer function given by eq.(6)[25], where K_3 is current driver constant. The output of the actuator is the fuel-flow $\psi(s)$ and the input current is $I(s)$ [25,26].

$$\Psi (s) = \frac{K_3 K_2}{(1 + \tau_2 s)} I (s) \tag{6}$$

Fuel Flow $\Psi (s)$ is then converted into mechanical torque $T (s)$. The objective of the control system is to maintain the system's frequency at the desired reference value i.e. drives the frequency error ($f_m - f_r$) to zero where f_r is the reference frequency (50/60 Hz or 1 p.u.) and f_m is the measured frequency. ρ is the viscous friction coefficient [24,25]. Fig. 2. shows LFC model of diesel engine based generating unit.

3. Performance estimation of PID controller

In general, the PID controller design method using the integrated absolute error (*IAE*), or the integral of time multiplied by absolute error, or the integral of squared-error (*ISE*), or the integrated of time-weighted-squared-error (*ITSE*), is often employed in control system design because it can be evaluated analytically in the frequency domain [19,27]. The *IAE*, *ISE*, and *ITSE* performance criterion formulas are as follows:

$$IAE = \int_0^{\infty} |r(t) - y(t)| dt = \int_0^{\infty} |e(t)| dt \tag{7}$$

$$ITAE = \int_0^{\infty} t |e(t)| dt \tag{8}$$

$$ISE = \int_0^{\infty} e^2(t) dt \tag{9}$$

$$ISTE = \int_0^{\infty} t e^2(t) dt \tag{10}$$

4. Particle Swarm Optimization (PSO)

PSO is derived from the social-psychological theory, and has been found to be robust in complex systems. The PSO algorithm has been successfully applied to solve various optimization problems [19, 27]. After any iteration, all particles update their positions and velocities to achieve better fitness values according to the following:

$$v_{j,g}^{(t+1)} = w * v_{j,g}^{(t)} + c_1 * rand_1 * (pbest_{j,g} - x_{j,g}^{(t)}) + c_2 * rand_2 * (gbest_g - x_{j,g}^{(t)}) \tag{11}$$

$$x_{j,g}^{(t+1)} = x_{j,g}^{(t)} + v_{j,g}^{(t+1)} \quad j = 1, 2, \dots, n \quad g = 1, 2, \dots, m$$

where:

t is the current iteration number, $pbest_j$ is $pbest$ of particle j , $gbest_g$ is $gbest$ of the group, $rand_1$; $rand_2$ are two random numbers in the interval $[0, 1]$, c_1 ; c_2 are positive constants and w parameter for controlling the impact of previous velocities is commonly known as the inertia weight. It influences the tradeoff between the global and local exploitation abilities of the particles. The inertia Weight is modified as:

$$\omega = \omega_{max} - \left(\frac{\omega_{max} - \omega_{min}}{iter_{max}} \right) itre \tag{12}$$

where ω_{min} , ω_{max} , $iter$, and $iter_{max}$ are minimum, maximum values of ω , the current iteration number and pre-specified maximum number of iteration cycles, respectively [19,27].

5. Fuzzy Based PID Controller

5.1 There are three principal elements to a fuzzy logic controller:

Fuzzy logic controller working on crisp information involves various stages to exploit fuzzy inference mechanism. Fuzzification Stage (Fuzzifier), Rule base and Inference engine stage and Defuzzification stage (Defuzzifier). For Load Frequency Control the process operator is assumed to respond to variables error (\mathcal{E}) and change of error ($c\mathcal{E}$) as shown in fig.3 [28-30].

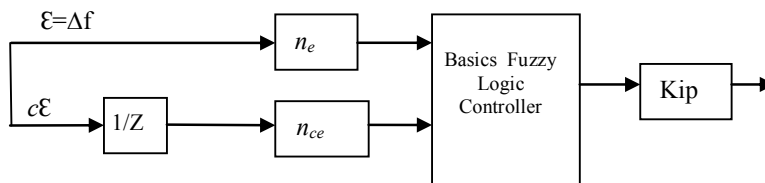


Fig. 3. Fuzzy Logic Controller (FLC).

The variable error is equal to the real power system frequency deviation ($d\hat{f}_i$). The frequency deviation ‘ f ’ is the difference between the nominal or scheduled power system frequency (f_N) and the real power system frequency (f). Taking the scaling gains into account, the global function of the FLC output signal can be written as.

$$dP_c = F[k_e \mathcal{E}(k), k_{ce} c\mathcal{E}(k)] \tag{13}$$

Where, n_e and n_{ce} are the error and the change of error scaling gains, respectively, and F is a fuzzy nonlinear function. FLC is dependant to its inputs scaling gains.[28-30] The block diagram of FLC is shown in fig.3. A label set corresponding to linguistic variables of the input control signals, $\mathcal{E}(k)$ and $c\mathcal{E}(k)$, with a sampling time of 0.01 sec is as follows[28]:

$$L(\mathcal{E}, c\mathcal{E}) = \{BN, MN, SN, ZE, SP, MP, BP\} \tag{14}$$

Where, BN = Big Negative ; MN = Medium Negative ; Small Negative (SN) ZE = Zero; SP=Small Positive ; MP = Medium Positive ; BP = Big Positive. Fuzzy logic controller has been used in three control areas comprised of hydro-thermal and three areas of DFIG-Thermal-Diesel inter connected in a ring. Attempt has been made to examine with five number of triangular membership functions (MFs) which provides better dynamic response with the range on input (error in frequency deviation and change in frequency deviation) i.e. universe of discourse is -1 to 1. The number of rules are 56. The results obtained are compared with the existing one and shown to prove better. In addition, several inputs have been tried out and dynamic responses are examined in order to decide suitable inputs to the fuzzy logic controller (FLC). Linguistic Rules for proportional gain K for $\mathcal{E}(k)$ and $c\mathcal{E}(k)$ [28,29]. Linguistic Rules for proportional gain K for $\mathcal{E}(k)$ and $c\mathcal{E}(k)$

$\mathcal{E}(k)$ $c\mathcal{E}(k)$	BN	MN	SN	ZE	SP	MP	BP
BN	BN	BN	BN	MN	SN	SN	ZE
MN	BN	MN	MN	MN	SN	ZE	SP
SN	BN	MN	SN	SN	ZE	SP	MP
ZE	BN	MN	SN	ZE	SP	MP	BP
SP	MN	SN	ZE	SP	SP	MP	BP
MP	SN	ZE	SP	MP	MP	MP	BP
BP	ZE	SP	SP	MP	BP	BP	BP

$\mathcal{E}(k)$ $c\mathcal{E}(k)$	BN
BN	BN
MN	SN
SN	MN
ZE	ZE
SP	SP
MP	MP
BP	BP

For integral gain I where K is BN, Fuzzy logic controller is implemented to improve the dynamic performance of interconnected system.

6. Five area LFC (load frequency control) model under study

Figure 6 shows simulink based test model of five area frequency model. The performance of PI controller based on fuzzy rules and PSO has been carried out for different operating conditions given in the next sections as follows.

6.1 LFC control with the help of conventional PI controller in main grids and I controller in micro-grids with arbitrary selected values.

For an individual participation of each GU will act as local demand to each CA. A five-control area power system shown in fig. 4 is considered as a test system. It is assumed that each main grid control area includes two Gencos and one Disco and each micro Grid control area include three gencos (Wind turbine, Diesel Generator and a Thermal plant). All the five areas are connected to form a ring configuration. The power system parameters are tabulated in Tables 1, 2 and in appendix A.

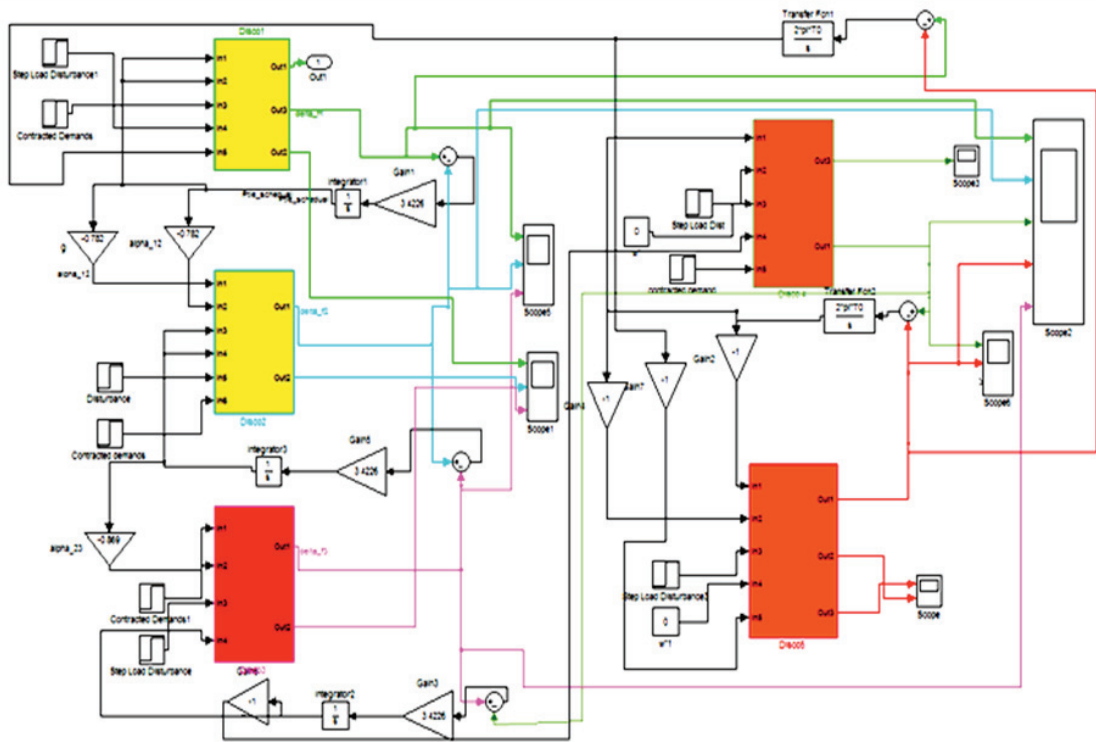


Fig. 4. Five Control Area Simulink Test Model

Table 1. Applied data for gencos of three Main grids[1]

Quantity	Gen 1	Gen 2	Gen 3	Gen 4	Gen 5	Gen 6
Rating (MW)	800	1000	1100	1200	1000	1000
R(Hz/pu)	2.4	3.3	2.5	2.4	3	2.4
Tt(s)	0.36	0.42	0.44	0.4	0.36	0.4
Tg(s)	0.06	0.07	0.06	0.08	0.07	0.08
Alpha	0.5	0.5	0.5	0.5	0.5	0.5

Table 2. Area wise data[1]

Quantity	Area1	Area2	Area3	Area4	Area5
D(pu/Hz)	0.0084	0.014	0.01	0.0161	0.0161
2H(pu)	0.1667	0.2	0.1667	0.1612	0.1612
B(pu/Hz)	0.8675	0.795	0.870	0.3483	0.3483
K(s)	0.2695	0.0418	0.2319	0.2695	0.2695
	-0.3788/s	-0.1806/s	-0.3796	-0.3788/s	-0.3788/s
				0.05	0.05
Tij			0.545		

6.1.1 Scenario-regulated environment

It is assumed that a step increase in demand as $dPL_1 = 100MW$, $dPL_2 = 70MW$, $dPL_3 = 60MW$, $dPL_4 = 20MW$ and $dPL_5 = 10MW$ are applied to the control areas and each CA demand is sent to its local GU only, based on the following GC as given eq.(15) for a regulated system.

$$GC = \begin{bmatrix} 0.5 & 0 & 0 & 0 & 0 \\ 0.5 & 0 & 0 & 0 & 0 \\ 0 & 0.5 & 0 & 0 & 0 \\ 0 & 0.5 & 0 & 0 & 0 \\ 0 & 0 & 0.5 & 0 & 0 \\ 0 & 0 & 0.5 & 0 & 0 \\ 0 & 0 & 0 & 0.33 & 0 \\ 0 & 0 & 0 & 0.33 & 0 \\ 0 & 0 & 0 & 0.33 & 0 \\ 0 & 0 & 0 & 0 & 0.33 \\ 0 & 0 & 0 & 0 & 0.33 \\ 0 & 0 & 0 & 0 & 0.33 \end{bmatrix} \dots\dots\dots(15)$$

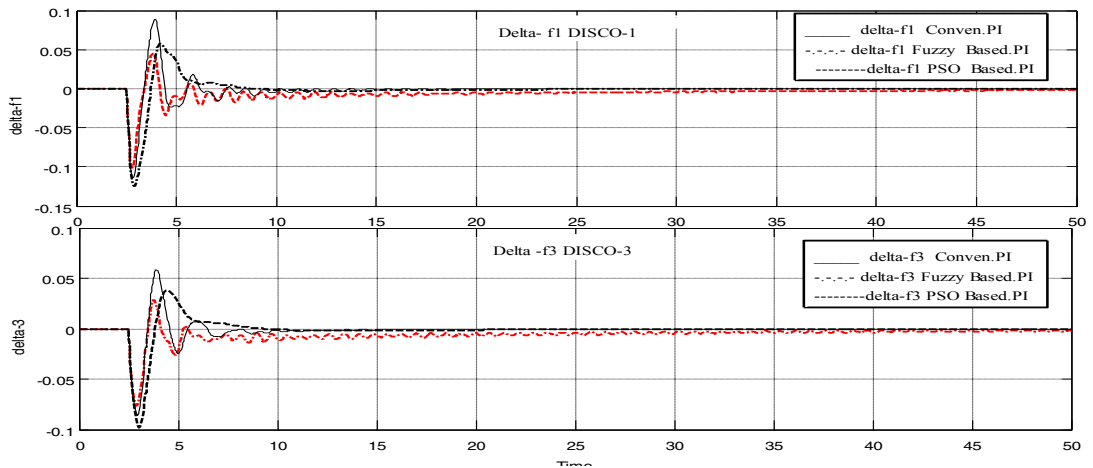


Fig. 5. (a) System response for frequency deviation

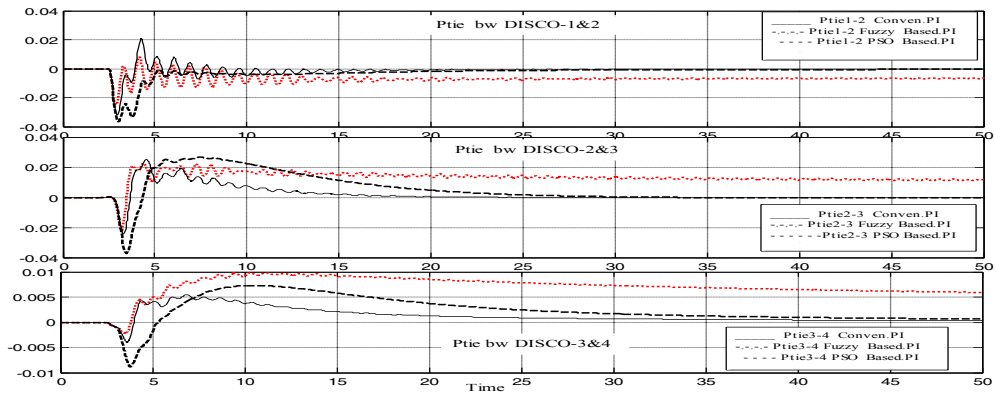


Fig. 5. (b) Changes occurred in Tie-line Powers

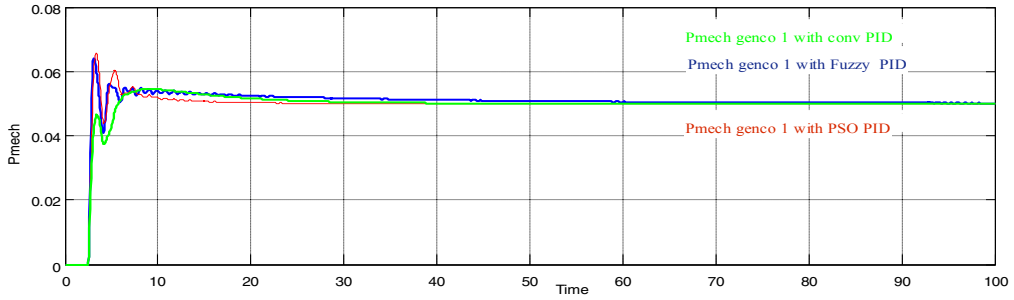


Fig. 5. (c) Changes occurred in Mechanical Powers

Table.3 Arbitrary Pre-Selected values of Conventional PI Gains with D=0 [1]

Discos	P	I
Disco 1	0.2695	-0.3788
Disco 2	0.0418	-0.1806
Disco3	0.2319	-0.3797
Disco 4	0	-0.05,-0.3788
Disco 5	0	-0.05,-0.3788

Table 4 Obtained Error Values with arbitrary PID values.

DISCOs and GENCOs	ISE	ITSE	IAE	ITAE	
DISCO 1 1,2	0.01176	0.04013	0.1786	0.8685	
DISCO 2 3,4	0.006143	0.02524	0.1834	1.167	
Disco 3 5,6,	0.005196	0.01803	0.1197	0.5444	
Disco 4 7,8	0.001426	0.008456	0.1348	2.052	
	9	0.003619	0.0125	0.1115	0.6622
Disco 5 10,11	0.00334	0.00052	0.0962	1.688	
	12	0.002158	0.006691	0.06766	0.3419

Table 5.Obtained Errors for PSO based optimised P=0.63, I=-0.1485, D=0

DISCOs	GENCOs	ISE	ITSE	IAE	ITAE
Disco 1	1,2	0.2303	0.08954	0.3638	2.603
Disco 2	3,4	0.01589	0.1002	0.4318	3.956
Disco 3	5,6	0.01421	0.05704	0.2672	1.816
Disco 4	7,8	0.004247	0.2177	0.6435	32.98
	9	0.02593	0.09716	0.3135	1.845
Disco 5	10,11	0.009276	0.05473	0.332	3.907
	12	0.04352	0.0181	0.5073	3.686

Table 6. Obtained Errors for Fuzzy based optimised P=0, I=0, D=0

DISCOs	GENCOs	ISE	ITSE	IAE	ITAE
Disco 1	1,2	0.005977	0.02062	0.1662	2.166
Disco 2	3,4	0.02585	1.148	1.518	75.4
Disco 3	5,6	0.002849	0.01335	0.168	4.234
Disco 4	7,8	0.007168	0.1964	0.7445	29.55
	9	0.004389	0.03694	0.3383	8.114
Disco 5	10,11	0.006146	0.2072	0.7304	31.09
	12	0.003916	0.03285	0.3315	7.734

The frequency deviation (Δf_i), tie-line power flow (ΔP_{tie}) and mechanical power changes[1] are shown in Figs. 5(a),(b) and(c) for the closed-loop system. Since there are no contracts between areas, the scheduled steady-state power that flows over the tie-lines is zero. Figure 5(a) shows the change in tie line power of each disco and variations in the tie line power are found to be so small with PSO technique compared to fuzzy and trail approach. Table 4,5 and 6 shows that errors produced in each disco which indicates the level of variations in the frequency deviations and tie line power changes.

6.1.2. Scenario de-regulatedEnvironment

It is assumed that a step increase in demand as $dPL1 = 100MW$, $dPL2 = 100MW$, $dPL3 = 50MW$, $dPL4 = 60MW$ and $dPL5 = 20MW$ are applied to the control areas and each Disco demand is sent to its local Gencos only, based on the following GC matrix as given eq.(16) for a regulated system. Now, there are contracts between areas, the scheduled steady-state power that flows over the tie-lines is deviates from its previous steady state zero level as shown in Fig. 6(b). The actual mechanical powers are shown in Fig. 6(c) .

$$GC = \begin{bmatrix} 0.3 & 0.1 & 0 & 0.2 & 0 \\ 0.3 & 0 & 0 & 0 & 0 \\ 0 & 0.1 & 0 & 0 & 0.25 \\ 0 & 0.6 & 0 & 0 & 0 \\ 0 & 0 & 0.2 & 0.1 & 0 \\ 0.1 & 0.2 & 0.6 & 0 & 0 \\ 0.3 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0.1 \\ 0 & 0 & 0.2 & 0 & 0 \\ 0 & 0 & 0 & 0.1 & 0 \\ 0 & 0 & 0 & 0.25 & 0.35 \\ 0 & 0 & 0 & 0.35 & 0.3 \end{bmatrix} \dots\dots\dots(16)$$

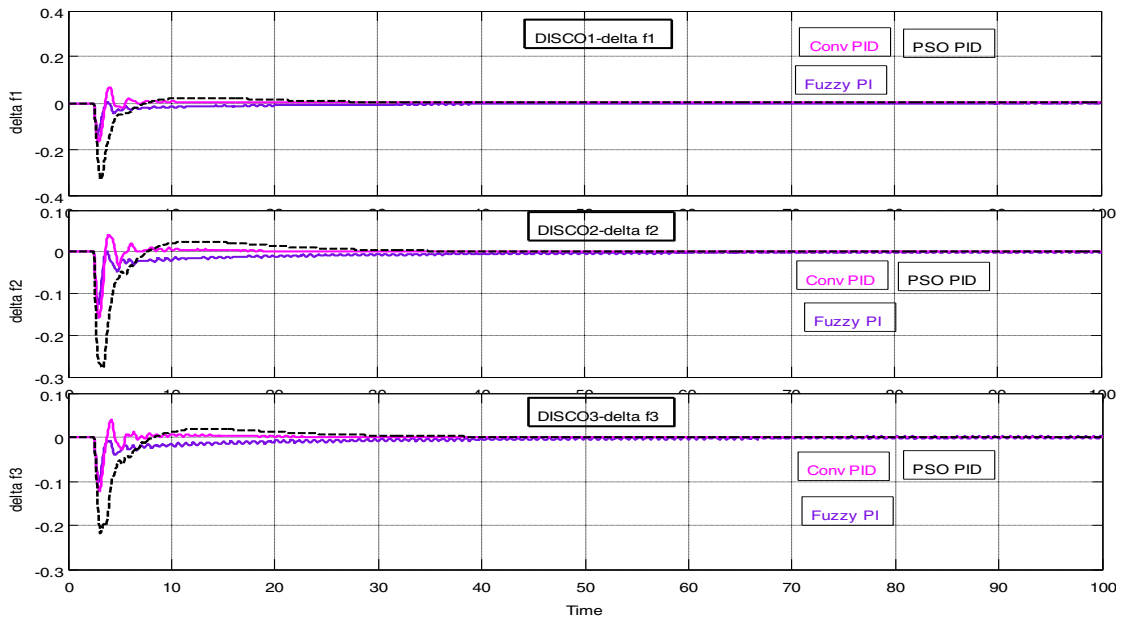


Fig. 6(a). System response for frequency deviation

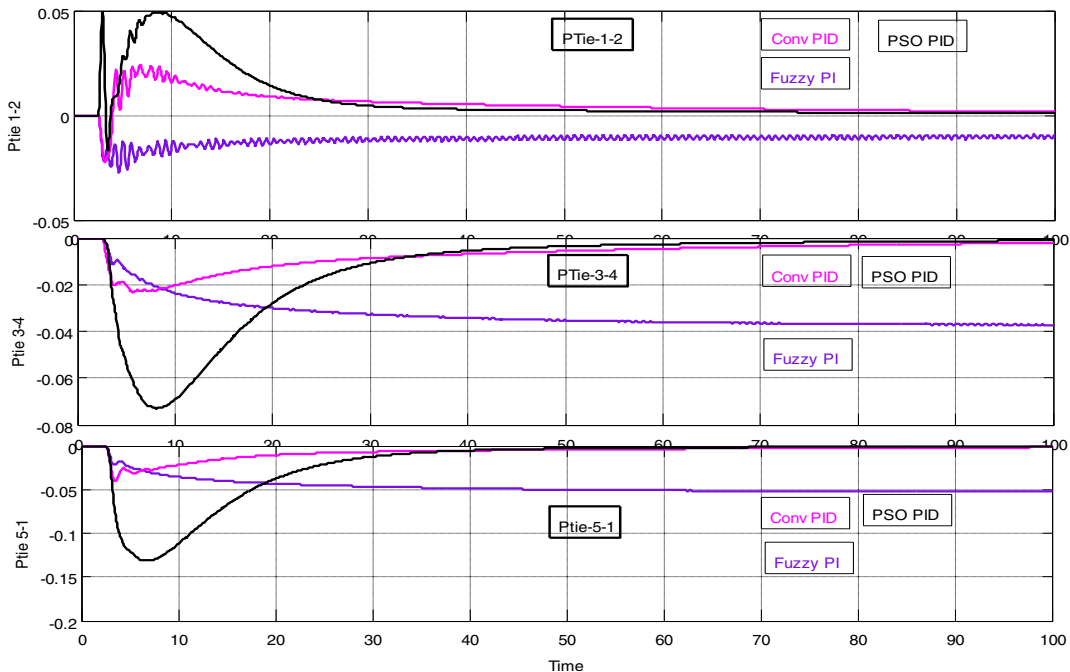


Fig. 6(b).Changes occurred in Tie-line Powers

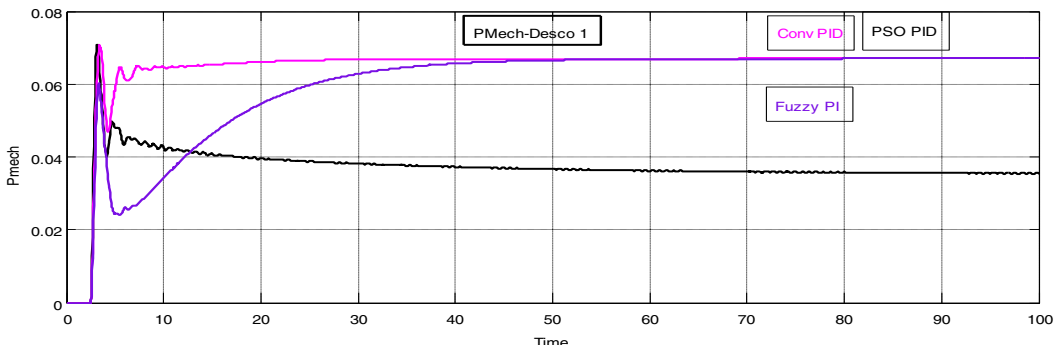


Fig. 6(c).Changes occurred in Mechanical Powers

Table 7. Obtained Error Values with arbitrary PID values.

DISCOs and GENCOs		ISE	ITSE	IAE	ITAE
DISCO 1	1,2	0.0228	0.07325	0.2205	0.9328
DISCO 2	3,4	0.03409	0.1596	0.5261	3.841
Disco 3	5,6,	0.005843	0.02109	0.1421	0.8165
Disco 4	7,8	0.006412	0.2355	0.7363	31.42
	9	0.005332	0.01898	0.1633	1.661
Disco 5	10,11	0.00845	0.1689	0.7238	24.611
	12	0.0008746	0.00322	0.06074	0.611

Table 8. Obtained Errors for PSO based optimised P=0.63, I=-0.1485, D=0

DISCOs	GENCOs	ISE	ITSE	IAE	ITAE
Disco 1	1,2	0.1743	0.8428	1.309	11.58
Disco 2	3,4	1.04	7.114	3.87	36.25
Disco 3	5,6	0.0229	0.09032	0.3414	2.514
Disco 4	7,8	0.1733	1.408	1.813	28.42
	9	0.03718	0.1736	0.5857	6.459
Disco 5	10,11	0.05181	0.6443	1.266	26.08
	12	0.04086	0.04086	0.2677	2.538

Table 9. Obtained Errors for Fuzzy based optimised P=-0.04, I=0, D=0

DISCOs	GENCOs	ISE	ITSE	IAE	ITAE
Disco 1	1,2	0.4103	20.42	6.258	319.3
Disco 2	3,4	0.2368	11.92	4.768	243.7
Disco 3	5,6	0.006867	0.01996	0.5913	26.63
Disco 4	7,8	0.01583	1.036	1.157	70.16
	9	0.01059	0.116	0.6112	15.68
Disco 5	10,11	0.1004	6.051	3.044	172.9
	12	0.006015	0.07706	0.5087	12.97

The frequency deviation (Δf_i), tie-line power flow (ΔP_{tie}) and mechanical power changes [1] are shown in Figs. 6(a) and 6(b) for the closed-loop system. Table 7, 8 and 9 shows the error obtained for contracted demands according to eq.16 with arbitrary, PSO optimized and Fuzzy based PID gains.

7. Conclusion

From the above listed case studies it has been concluded that all the five areas work very well if the GC matrix given in the equation (15) is used i.e. raised demand in a disco is sent to its local GUs. These error values actually decide the area control error input signal to the PID controller unit whose main function is to minimize the error at the output. Hence, from the above discussion, PSO technique gives much improved results than fuzzy and trail methods because fuzzy technique requires large execution time to execute large no. of linguistic rules in a multi-area system at same time. Secondly, requirement of correct linguistic rule matrix to reach desired results is a time consuming process. In a trial method, to reach perfect values of PID gains, there is need of large no. of iterations.

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Appendix-A Data for Micro Grids

He1=3.5;PU.MW.secHe2=3.5;PU.MW.sec, Kgc1=0.05;Kgc2=0.05;Kp1=62; HZ/PU Ki1=0.05; Ki2=0.05; Kwi1=0.1; Kwp1=1.61; Kp2=62; HZ/PU Kwi2=0.1; Kwp2=1.61; B1=0.3483; B2=0.3827; R1=3; Hz/PU.MW R2=3; Hz/PU.MW T0=0.07; PU.MW/Hz Ta1=0.2; Sec Ta2=0.2;Sec Th1=0.1; Sec Th2=0.1; Sec,Tp1=10; sec,Tp2=10; sec, Tr1=0.1; Sec, Tr2=0.1; Sec, Tt1=1; Sec, Tt2=1; Sec, Tw1=6; Sec Tw2=6; Sec