



Alexandria University
Alexandria Engineering Journal

www.elsevier.com/locate/aej
www.sciencedirect.com



REVIEW

Economical analysis of combined fuel cell generators and absorption chillers

M. Morsy El-Gohary

Naval Arch. & Marine Eng. Dept., Faculty of Engineering, Alexandria University, Egypt

Received 15 August 2011; revised 16 December 2012; accepted 26 December 2012

Available online 1 February 2013

KEYWORDS

Economical aspects;
Co-generation system;
Fuel cell;
Absorption system

Abstract This paper presents a co-generation system based on combined heat and power for commercial units. For installation of a co-generation system, certain estimates for this site should be performed through making assessments of electrical loads, domestic water, and thermal demand. This includes domestic hot water, selection of the type of power generator, fuel cell, and the type of air conditioning system, and absorption chillers. As a matter of fact, the co-generation system has demonstrated good results for both major aspects, economic and environmental. From the environmental point of view, this can be considered as an ideal solution for problems concerned with the usage of Chlorofluoro carbons. On the other hand, from the economic point of view, the cost analysis has revealed that the proposed system saves 4% of total cost through using the co-generation system.

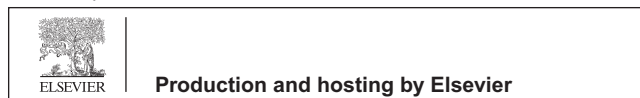
© 2013 Faculty of Engineering, Alexandria University. Production and hosting by Elsevier B.V.
All rights reserved.

Contents

| | |
|---|-----|
| 1. Introduction | 152 |
| 1.1. Fuel cell description | 152 |
| 2. The concept of co-generation | 153 |
| 3. Natural gas as a heat source for fuel cell and absorption chillers | 153 |
| 4. Absorption cooling systems | 154 |
| 5. Cost analysis | 155 |
| 5.1. Initial costs | 155 |
| 5.2. Operating costs | 156 |

E-mail address: prof.morsy@gmail.com

Peer review under responsibility of Faculty of Engineering, Alexandria University.



Nomenclature**Abbreviations**

| | |
|-------|---------------------------------|
| AAC | average annual cost |
| AOC | annual operating cost |
| CFC | chlorofluorocarbon |
| Cg | co-generation |
| CHP | combined heat and power |
| EPA | environmental protection agency |
| FC | fuel cell |
| HCFC | hydro chlorofluorocarbon |
| Hex | heat exchanger |
| HFC | hydro fluorocarbon |
| HG | high grade |
| HRC | heat rejection cost |
| i | annual interest rate |
| I | initial cost |
| kW | kilo watt |
| L.F | load factor |
| LG | low grade |
| MCFC | molten carbonate fuel cell |
| NGS | natural gas system |
| P_c | present cost |
| PAFC | phosphoric acid fuel cell |

| | |
|-------|------------------------------------|
| PEMFC | proton exchange membrane fuel cell |
| PNHR | plant net heat rate |
| SOFC | solid oxide fuel cell |
| TAC | total annual cost |
| US | USA dollar |
| VAS | vapor absorption system |
| VCS | vapor compression system |

Subscripts

| | |
|-----------------|--------------------------|
| C | initial cost |
| CO ₂ | carbon dioxide |
| I | irreversibility |
| j | at month j of the year |
| K | at hour k of the day |

Superscripts

| | |
|-----|--------------------------|
| j | at month j of the year |
| K | at hour k of the day |
| m | monthly |
| n | number of days per month |

| | | |
|------|--------------------------------|-----|
| 5.3. | Maintenance cost | 156 |
| 5.4. | Average annual cost comparison | 157 |
| 6. | Discussion of results | 157 |
| 7. | Conclusion | 158 |
| | References | 158 |

1. Introduction

Co-generation is the simultaneous production of electricity and thermal energy from the same fuel source. In fact, it can be applied to any commercial, industrial, or institutional facility where there is a simultaneous need for both heat energy and electrical power. Actually, it offers several advantages over central electricity generating stations:

1. Fuel use efficiency is much higher, often twice as high, since the rejected heat is normally utilized in a useful process or hot water heating.
2. Because of its higher efficiency, co-generation is also economically and financially more attractive than central power generating stations.
3. Much, if not most, of the cogenerated electricity is consumed at the generation site, thus saving transmission line capacity and costs.
4. Unlike central power generating stations, co-generation can be cost-effective even in very small capacities (as low as 50–100 kW).
5. Most co-generation projects have a much shorter lead time than the large central generating stations, as shown in Fig. 1 [1].

1.1. Fuel cell description

Fuel cells are the electrochemical devices which convert the chemical energy of a reaction directly into electrical energy. The basic physical structure or building block of a fuel cell consists of an electrolyte layer in contact with a porous anode and cathode on either side.

A schematic representation of a fuel cell with the reactant/product gases and the ion conduction flow directions through the cell is illustrated in Fig. 2 [1].

Fuel cells are named or defined according to their electrolyte, i.e., phosphoric acid, molten carbonate, solid oxide, or proton exchange membrane. During the last 50 years, a number of diverse types of fuel cells have been developed. At present, the most dominant type is the Proton Exchange Membrane Fuel Cell (PEMFC). In fact, the PEMFCs are the most versatile of all fuel cells and, depending on their size (ranging from less than 1 W to 300 kW), they can propel anything from small electronic devices to buses and submarines.

However, other fuel cells, though they have their own advantages, are less flexible. For example, Molten Carbonate Fuel Cells (MCFCs) are only used in stationary applications (250 kW and above), like Phosphoric Acid Fuel Cells (PAFCs)

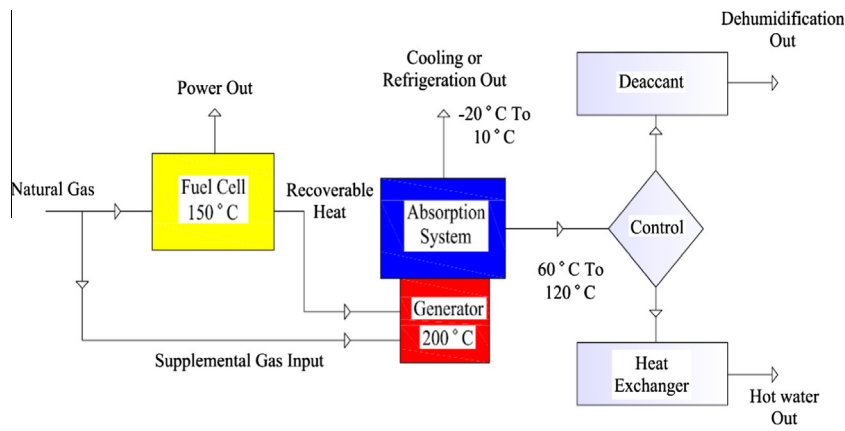


Figure 1 Cogeneration station.

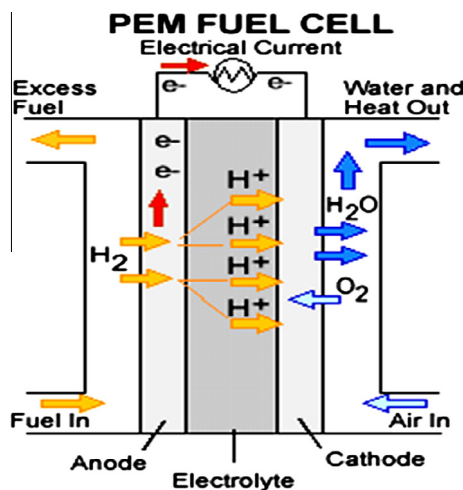


Figure 2 Schematic of an individual fuel cell.

(50 kW and above). Moreover, Direct Methanol Fuel Cells (DMFCs) are best suited for the applications where less than 1 kW is required. Furthermore, Solid Oxide Fuel Cells (SOFCs) are more flexible (1 kW to several 100 kW), but they are not, as the technology now stands, suitable for use in vehicles.

2. The concept of co-generation

In a central-station steam thermal plant, fuel is burned to produce high-temperature and high pressure steam, which are then passed through a turbine to generate electricity. However, even in the most efficient of these plants, less than 40% of the available energy (heat) contained in the fuel is converted into electricity while the remainder is lost in the atmosphere. Some of the energy loss is accounted for by combustion gases that escape to the atmosphere through the boiler stack. Nevertheless, most of the energy is wasted in the condensation and cooling of steam and water after they have passed through the turbine.

Fig. 3 illustrates the basic idea of co-generation; yet, a large number of variations on this matter are possible. All of them combine the generation of a mechanical or electrical power through the utilization of waste heat. Generally speaking, the term co-generation is widely used and universally accepted to

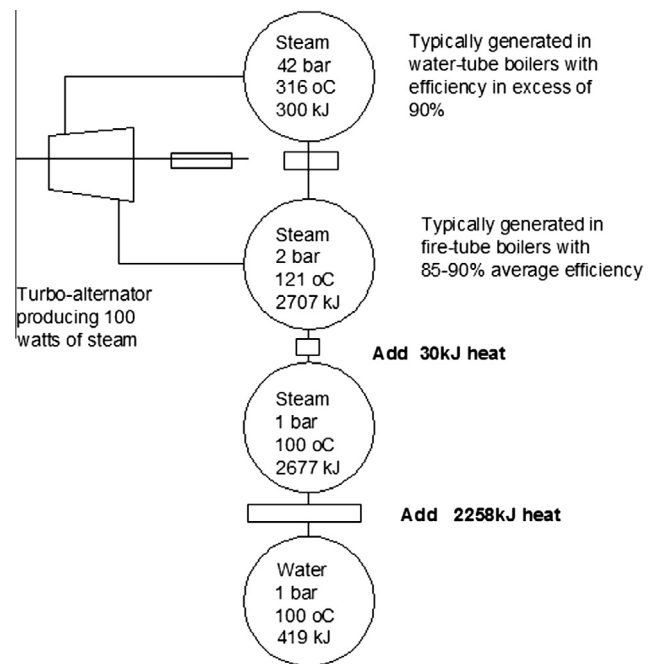


Figure 3 The energy requirement for cogeneration.

describe both the concept of the combined production of power and heat as well as the equipment or systems utilized to produce power and heat in this way. Another form commonly applied to such systems and equipment, especially in Europe, combines both heat and power, often referred to in the literature as CHP [2-4].

3. Natural gas as a heat source for fuel cell and absorption chillers

The availability of a new generation of more efficient and reliable gas cooling products from a number of manufacturers is only one reason for the renewed interest in gas cooling. Other recent developments which contribute to the momentum toward natural gas cooling include the following:

- The natural gas clean environmental effect through CFC free technology.

Table 1 Electric daily load demand.

| Daily load (W h/day) | Peak load (We) | Min load (We) | <i>n</i> (units) | Peak load/ <i>n</i> (We) |
|----------------------|----------------|---------------|------------------|--------------------------|
| 22,445,252 | 1969799.038 | 644166.688 | 2 | 984899.519 |

Table 2 Site capacity degradation.

| Degradation | % |
|---|------------|
| Temp. capacity degradation | 0 |
| Altitude capacity degradation | 0 |
| De-rating until overhaul (fuel processor and stack change) | 16 |
| Auxiliaries consumption | 4.2 |
| Combined degradation factor | 1.24047619 |
| Actual rating for unit (watt) | 1,222,000 |

- The desire to cut energy costs and eliminate peak electric demand charges.
- The financial incentives from the gas industry.
- The need for improved indoor air quality.
- The low natural gas prices [5,6].

Standalone combined heat and power is based on electrical load which consist of two fuel cell generators and absorption chiller, while natural gas is used as a heat source for fuel cell and absorption system [7].

In this case, selection of fuel cells is based on electrical load demand by taking into consideration the maximum load and minimum load according to the chronological load curve for the case study. For example, if we select 1FC for maximum load and minimum load less than 10–15%, this FC will not work as shown in Tables 1 and 2. Thus, optimization of the selection of fuel cell number should be performed in order to meet this load. Therefore, the following Figs. 4–6 illustrate the selection of size on the seasons. Moreover, there is the site capacity degradation which affects the performance of a generator [2,8].

A theoretical analysis was completed of the electrical load in summer for two different systems: the peak load of a fuel

cell and the peak load for two fuel cells relative to time, as shown in Fig. 4.

A theoretical analysis was conducted of the electrical load in winter for two different systems: the peak load of one fuel cell and the peak load of two fuel cells relative to time, as illustrated in Fig. 5.

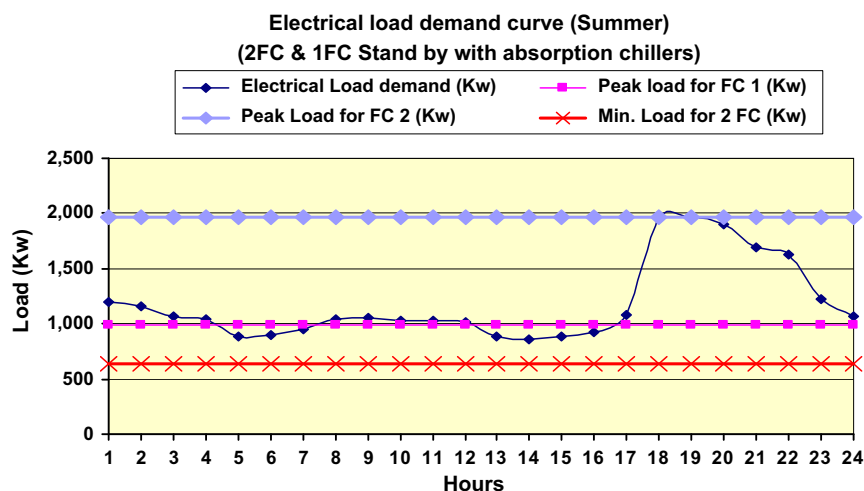
A theoretical analysis was done of the electrical load in Autumn and Spring for two different systems: the peak load of one fuel cell and the peak load of two fuel cells relative to time, as demonstrated in Fig. 6.

4. Absorption cooling systems

It has been on the market for over 100 years. In the late 1800s, absorption chillers were used for large refrigeration plants. However, during the 1950s, technological advances occurred, and the systems were fine-tuned for commercial use. Conversely, their popularity declined in the late 1970s due to the inexpensive cost and abundance of electricity [9,10].

Similar to the vapor compression-cycle, absorption chillers rely on a cycle of condensation and evaporation to produce cooling. Both systems have an evaporator and coil condenser which expands the refrigerant from high to low pressure between the condenser and evaporator. The mechanical compressor of the vapor-compression cycle of condensation and evaporation is replaced by a heat source in the absorption chiller. This heat source is either directly fired using a burner or indirectly fired using steam, hot water, or waste heat from other processes. Most absorption systems use a water and lithium bromide combination as working fluid [11,12].

Absorption chillers are available in two types. The first, a single-effect, is operated with low-grade waste heat. On the other hand, the second, the double-effect, requires either direct firing or high-grade steam as the heat source [9,13].

**Figure 4** Electric load demand curve in summer relative to time.

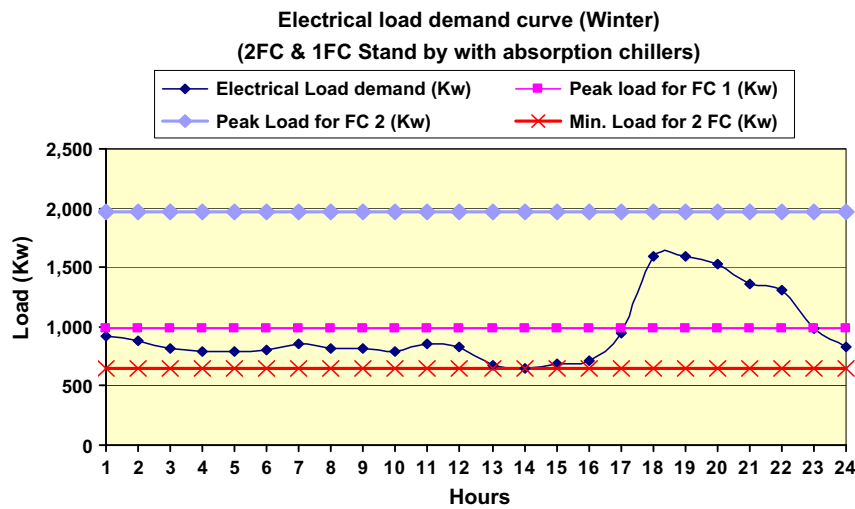


Figure 5 Electric load demand curve in winter relative to time.

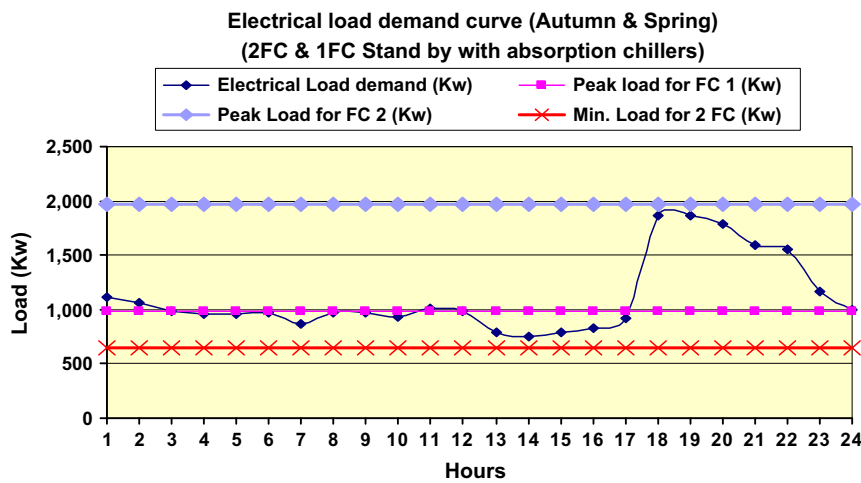


Figure 6 Electric load demand curve in Autumn and Spring relative to time.

5. Cost analysis

An initial estimate of the costs must be computed in order to determine the economic feasibility of any project.

Therefore, there is a method to compare the cost of any two or more systems. That is the Life-Cycle Cost (LCC), which includes all cost factors (first cost, operating cost, maintenance, replacement and estimated energy use) and can be used to evaluate the total cost of the system over the complete life of the system.

Thus, the data of the required costs presented in this study produce a very good estimate of the capital costs and operating costs. Eq. (1) confirms that the total cost of any system consists of three main variables, namely the initial cost, the operating cost, and finally the maintenance cost.

$$\text{Total cost} = f(\text{initial cost} + \text{operating cost} + \text{maintenance cost})$$

$$\text{Total cost} = \sum_{k=1}^k I_k + \sum_{j=1}^{12} (I + i_m)^{(12-j)} * \sum_{k=1}^k O_{j,k}^m + \sum_{k=1}^k M_k \quad (1)$$

The following section presents details of how these various costs are evaluated [7,14,15].

5.1. Initial costs

The initial costs for the single-effect vapor absorption systems comprise the absorption machine, heat rejection equipment, and natural gas system.

The initial cost of the vapor compression system includes the vapor compression chiller and the heat rejecting equipment. The physical size of the absorption system is larger than the size of the vapor compression system. Consequently, this increase in size requires a larger building, moving equipment, and support systems. Consequently, this results in a higher installation cost for the vapor absorption system.

Furthermore, the electric supply for a vapor compression system needs to be upgraded by either increasing the electrical capacity of the electric substation or building a new substation.

The initial cost, therefore, should include, in addition to the purchase and installation expenses of the systems, the various subsystems necessary for an effective operation. This

Table 3 Initial cost of the two systems.

| Initial cost | | Single-effect VAS | Vapor comp. system |
|--|--------|-------------------|--------------------|
| <i>Absorption chillers and VC system</i> | | | |
| Machine capacity (kW) | 200 kW | | |
| Total cost (\$) | 59,000 | 43,500 | |
| Life time (year) | 20 | 10 | |
| <i>Heat rejection equipment</i> | | | |
| Total cost (\$) | 26,000 | 17,250 | |
| Life time (year) | 20 | 20 | |
| <i>Natural gas system</i> | | | |
| Cost (\$) | 15,000 | 0 | |
| Life time (year) | 20 | 0 | |

comprises piping, wiring, and specific structures. Table 3 shows the initial cost of both systems.

In addition, the heat rejection equipment for the application considered in this work is a cooling tower. The cooling towers for both vapor absorption systems (single and double-effect) have a centrifugal or propeller type fan. Although the centrifugal fan has a higher capital cost, it has been selected for two practical reasons: the lower level of noise, as well as its lower operating costs compared to the propeller fan.

The cooling tower for the vapor absorption system is between one and a half and two times larger than that for a vapor compression system of a similar size.

The costs of the auxiliary equipment cover the electric pump motors, fan motors, and a water treatment system for the cooling tower [8,16]. The cost of natural gas auxiliary is zero because it does not contain a natural gas system.

5.2. Operating costs

Operating costs, which encompass the costs of electricity, wages of employees, supplies, water and materials, are those incurred by the actual operation of the system [8,16]. All new plants may be assumed to be fully automatic. Thus, estimation of labor is very rarely relevant. For the purpose of comparison between the two systems and taking into account the low labor cost in Egypt, the assumption of zero difference in the operating labor cost for the two systems is made. Hence, the electrical operating costs for the vapor absorption systems comprise the heating water pump, condenser water pump, generator water pumps and cooling tower fans:

$$O_{(VAS)}^m = n_m^* EC^* \sum_{k=1}^k E_k \quad (2)$$

In regard to the vapor compression system, the operating costs are dominated by the electricity required to drive the compressor. Additional electricity is used to drive the condenser water pump and the cooling tower fans. The annual operating cost can be displayed in Table 4. The monthly electric operating cost of the vapor compression system is proportional to its monthly electric energy consumption [8]:

$$O_{(VCS)}^m = n_m^* EC^* \int_{k=1}^{24} \frac{Q_k}{COP_k} \cdot dt \quad (3)$$

A theoretical analysis was conducted of the different cooling capacities between 200 and 800 kW and the annual initial

Table 4 Annual operating cost.

| Operating costs | | |
|-------------------------------------|---------|---------|
| Price of kW h (\$/kW h) | 0.08 | |
| VAS and VCS machines | VAS | VCS |
| Total use (kW h/year) | 34,840 | 536,930 |
| Annual cost (\$) | 3026 | 42,954 |
| Price of kW h (\$/kW h) | 0.06 | – |
| Natural gas system | VAS | VCS |
| Total use (kW h/year) | 265,536 | – |
| Annual cost (\$) | 15,932 | – |
| <i>Cooling water pump</i> | | |
| Cooling water pump motor efficiency | 0.68 | |
| Total use (kW h/year) | 56,290 | 42,530 |
| Annual cost (\$) | 5104 | 3400 |
| <i>Cooling tower fans</i> | | |
| Fans efficiency | 0.6 | |
| Fan partial use factor | 0.4 | |
| Total use (kW h/year) | 50,715 | 33,810 |
| Annual cost (\$) | 4060 | 2705 |
| Total annual operating cost (\$) | 28,122 | 49,059 |

and operating costs of two different systems, namely, the vapor compression system and vapor absorption system which were calculated as shown in Fig. 7.

5.3. Maintenance cost

Maintenance cost is the final cost to be estimated for air conditioning systems. There are various levels of maintenance, which may be applied to building HVAC services. The three most common levels are the run-to-failure, the preventive and finally the predictive maintenance.

Maintenance cost is difficult to quantify because it depends on a large number of variables, such as, local labor rates, their experience, the age of the system and operating time.

The maintenance cost for the heat rejection subsystem tends to be higher for the VAS due to more rapid scaling; however, this could be offset by the maintenance cost of the VCS because it is a work-operated cycle.

Maintenance costs cited in various studies reveal that the maintenance costs of a vapor absorption system range from 0.6 to 1.25 times the maintenance costs of the vapor compression system [17].

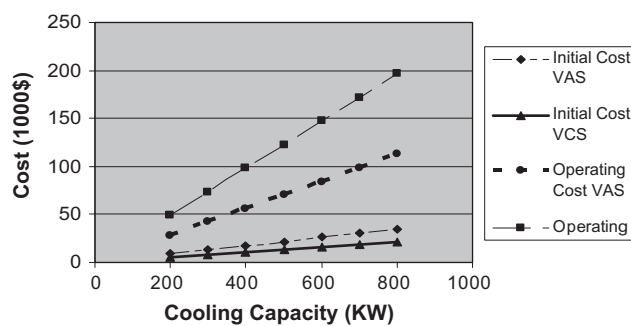


Figure 7 Initial and operating cost for VAS and VCS relative to cooling capacity.

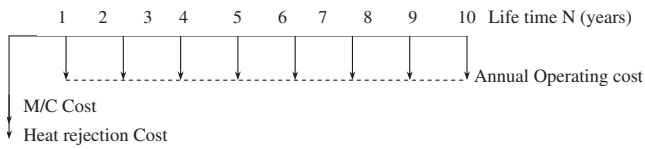


Figure 8 Cash flow diagram for VCS {AAC}.

5.4. Average annual cost comparison

With the average annual cost method, all costs occurring over a period are converted to an equivalent uniform yearly amount. Broadly speaking, the AAC comparison method is deemed as one of the most convenient methods, particularly for the systems composed of several subsystems with unequal life spans. Actually, this method does not require the assumption of replacement of a system. Fig. 8 represents the cash flow diagram for the VCS as there is no more assumption of the system replacement, more realistic in this case.

The AAC for the vapor absorption systems and the vapor compression system is the summation of the AAC values for the system and subsystems and the annual operating cost, Eqs. (4)–(7), respectively.

$$AAC_{VAS} = AA_{M/C} + AA_{HRC} + AA_{NGS} + AOC \quad (4)$$

$$AAC_{VCS} = AA_{M/C} + AA_{HRC} + AOC \quad (5)$$

$$AAC = P_C(A/P, i\%, N) \quad (6)$$

$$AAC = P_C \left[\frac{i(1+i)^N}{(1+i)^N - 1} \right] \quad (7)$$

6. Discussion of results

An analysis of the overall initial and operating costs for the two air conditioning systems have been developed in this work. This analysis describes two economic techniques for evaluating the systems. In addition, the work also pays special attention to cash flow over the complete life of the project (Life-Cycle Costing, LCC). The following Table 5 illustrates an equivalent annual cost comparison.

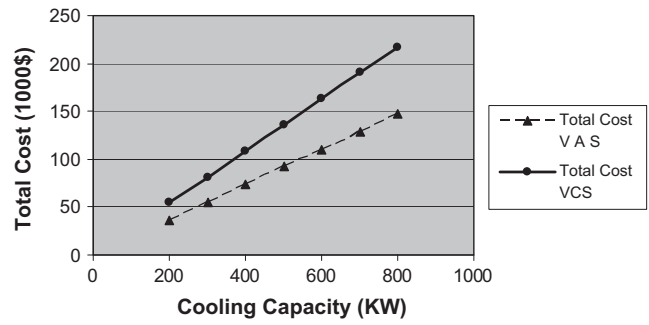


Figure 9 Total cost for VAS and VCS relative to cooling capacity.

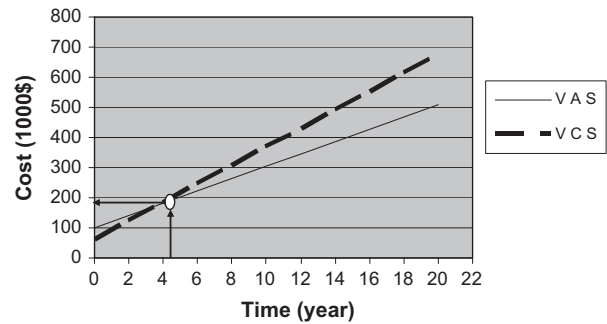


Figure 10 Total cost for VAS and VCS relative to time.

A theoretical analysis was performed of the different cooling capacities between 200 and 800 kW and the total cost of two different systems. These were the vapor compression system and vapor absorption system which were calculated and displayed in Fig. 9.

A theoretical analysis was conducted of the total cost of the two different systems: vapor compression system and vapor absorption system relative to time, as shown in Fig. 10.

The figure reveals that the initial cost of the VAS is higher than that of the VCS system; however, after about 4 years the break-even point occurs and the total cost of the VAS will be recommended to use because it will have a decreasing trend on the long run.

Table 5 The equivalent annual cost comparison results.

| Average annual cost comparison (AAC) | | |
|--|-------------------|--------|
| | VAS single-effect | VCS |
| Machine cost | 59,000 | 43,500 |
| Heat rejection cost | 26,000 | 17,250 |
| Natural gas system cost | 15,000 | 0 |
| AAC for machine cost | 5100 | 3792 |
| AAC for heat rejection | 2265 | 1503 |
| AAC for solar heat collection | 1307 | 0 |
| AAC for initial cost (1000 \$/year) | 8.672 | 5.295 |
| Annual operating cost AOC (1000 \$/year) | 28.122 | 49.059 |
| Annual interest rate | 0.06 | 0.06 |
| Life span (years) | 20 | 10 |
| Total average annual cost (1000 \$) | 36.794 | 54.885 |

7. Conclusion

Provided the co-generation is optimized in the way described above (i.e. sized according to the heat demand), the following benefits will emerge:

1. Increased thermal efficiency of energy conversion and co-generation use.
2. Lower emissions to the environment, in particular of CO₂, the main greenhouse gas.
3. In some cases, where there are biomass fuels and some waste materials, such as refinery gases, process or agricultural waste (either an aerobically digested or gasified), these substances can be used as fuels for co-generation schemes. Thus, they will lead to the increase in the cost-effectiveness and reduction of the need for waste disposal.
4. Large cost savings, providing additional competitiveness for industrial and commercial users, and presenting affordable heat for domestic users.
5. There would be available an opportunity to move towards more decentralized forms of electricity generation, where a plant is designed to meet the needs of local consumers, providing high efficiency, avoiding transmission losses and increasing flexibility in system use. This will particularly be the case if natural gas is the energy carrier.
6. Improved local and general security of supply – local generation, through co-generation, can reduce the risk that consumers may be left without supplies of electricity and/or heating.
7. An opportunity to increase the diversity of generation plant, and provide competition in generation. Co-generation provides one of the most important vehicles for promoting liberalization in energy markets.
8. From the economic point of view, the cost analysis has suggested that the proposed system saves 4% of total cost by using the co-generation system.

References

- [1] B. Roland, J. Schulte, H. Wendt, Phosphoric acid fuel cells – materials problems, process techniques and limits of the technology, in: *The International Fuel Cell Conference Proceedings*, NEDO/MITI, Tokyo, Japan, 1992.
- [2] M. Matsumoto, K. Usami, PAFC commercialization and recent progress of technology in Mitsubishi electric, in: *The International Fuel Cell Conference Proceedings*, NEDO/MITI, Tokyo, Japan, 1992.
- [3] M. Welaya, M.M. Elgohary, N. Ammar, Steam reforming fuel cells and other alternatives for marine electric power generation. *Alexandria Engineering Journal (AEJ)* AEJ, Elsevier Publications, 2011, pp. 69–75.
- [4] A.A. Manzela et al, Using engine exhaust gas as energy source for an absorption refrigeration system, *Journal of Power Sources* 87 (2010) 1141–1148.
- [5] *Natural Gas Cooling Equipment and Services Guide*, American Gas Cooling Center, Washington, DC, 1998.
- [6] M. Morsy El Gohary, The future of natural gas as a fuel in marine gas turbine for LNG carriers, *Proceedings of the Institution of Mechanical Engineers, Part M: Journal of Engineering for the Maritime Environment* 226 (4) (2012) 371–377.
- [7] *ASHRAE Handbook of Fundamentals*, American Society of Heating, Refrigeration and Air-conditioning Engineers, Atlanta, 1997.
- [8] *ASHRAE Handbook of HVAC Applications*, American Society of Heating, Refrigeration and Air-conditioning Engineers, Atlanta, 1999.
- [9] Egyptian Environmental affairs Agency, Central description of the environment in Egypt, Annex 5, Decree No. 338, 1995, pp. 10–68.
- [10] L.J. He, L.M. Tang, G.M. Chen, Performance prediction of refrigerant-DMF solutions in a single-stage solar-powered absorption refrigeration system at low generating temperatures, *Journal of Power Sources* 83 (2009) 2029–2038.
- [11] X.Q. Zhai, R.Z. Wang, Experimental investigation and performance analysis on a solar adsorption cooling system with and without heat storage, *Journal of Power Sources* 87 (2010) 824–835.
- [12] U. Eicker, D. Pietruschka, Design and performance of solar powered absorption cooling systems in office buildings, *Journal of Power Sources* 41 (2009) 81–91.
- [13] F. Agyenim, I. Knight, M. Rhodes, Design and experimental testing of the performance of an outdoor LiBr/H₂O solar thermal absorption cooling system with a cold store, *Journal of Power Sources* 84 (2010) 735–744.
- [14] EPA, Environmental Protection Agency, 40 CFR, Part No. 82, Protection of Stratospheric Ozone – Notice of Acceptability, Federal Register 61, No. 27, February 1996, pp. 39–47.
- [15] A.F. Elsafty, Investigation of water–lithium bromide vapour absorption air-conditioning system driven by solar energy. Ph.D. Dissertation. Coventry University CV14FP-UK, 2001.
- [16] *ASHRAE Handbook of Fundamentals*, American Society of Heating, Refrigeration and Air-Conditioning Engineers, Atlanta, 2005.
- [17] A.F. Elsafty, A.J. Al-Daini, Economical comparison between a solar powered vapor absorption air conditioning system and a vapor compression system in the middle east, *World Renewable Energy Congress* (2000) 569–583.