

TECHNICAL NOTE

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Effect of air flow on tubular solar still efficiency

Arunkumar Thirugnanasambantham^{1**}, Jayaprakash Rajan^{2*}, Amimul Ahsan² and Vinothkumar Kandasamy^{3*}

Abstract

Background: An experimental work was reported to estimate the increase in distillate yield for a compound parabolic concentrator-concentric tubular solar still (CPC-CTSS). The CPC dramatically increases the heating of the saline water. A novel idea was proposed to study the characteristic features of CPC for desalination to produce a large quantity of distillate yield. A rectangular basin of dimension $2 \text{ m} \times 0.025 \text{ m} \times 0.02 \text{ m}$ was fabricated of copper and was placed at the focus of the CPC. This basin is covered by two cylindrical glass tubes of length 2 m with two different diameters of 0.02 m and 0.03 m. The experimental study was operated with two modes: without and with air flow between inner and outer tubes. The rate of air flow was fixed throughout the experiment at 4.5 m/s. On the basis of performance results, the water collection rate was 1445 ml/day without air flow and 2020 ml/day with air flow and the efficiencies were 16.2% and 18.9%, respectively.

Findings: The experimental study was operated with two modes: without and with air flow between inner and outer tubes. The rate of air flow was fixed throughout the experiment at 4.5 m/s.

Conclusions: On the basis of performance results, the water collection rate was 1445 ml/day without air flow and 2020 ml/day with air flow and the efficiencies were 16.2% and 18.9%, respectively.

Keywords: Compound Parabolic Concentrator, Desalination, Tubular Solar Still

Introduction

Ahsan and Fukuhara [1] studied on a new heat and mass transfer tubular solar still and found that the heat and mass transfer coefficients can be expressed as functions of the temperature difference between the saline water and the cover. Ahsan et al. [2], experimentally studied on the evaporation, condensation and production of a tubular solar still and found that the relative humidity of the humid air was definitely not saturated and the hourly evaporation, condensation and production fluxes were proportional to the humid air temperature and relative humidity. Ahsan and Fukuhara [3-5] analyzed on a tubular solar still and on a quasi steady heat and mass transfer due to evaporation and condensation taking an account of humid air properties inside the still and found that the analytical solutions derived from the present model could reproduce the experimental results perfectly. Arunkumar et al. [6], studied on a hemispherical solar still.



Singh *et al.* [13], studied on the design parameters for concentrator assisted solar distillation system. Analytical expressions for the water and glass cover temperatures, the rate of heat flux due to evaporation, the rate of distillate output and the instantaneous thermal efficiency have been derived in terms of the system design and climatic parameters. It is analytically shown that the efficiency of the system with a concentrator was higher than that with a collector.

Chaochi *et al.* [14], designed and built a small solar desalination unit equipped with a parabolic concentrator and observed that the maximum efficiency corresponds to the maximum solar lighting obtained towards 14:00. At that hour, the boiler was nearly in a horizontal position, which maximizes the offered heat transfer surface. Mohamad and El-Minshawy [15] dealed with the status of solar energy as clean and renewable energy applications in desalination.



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^{*} Correspondence: tarunkumarsolar@gmail.com

Deceased

¹Department of Physics, Dr. NGP Institute of Technology, Coimbatore 641 048 Tamilnadu, India

Full list of author information is available at the end of the article

A novel idea proposed in this work is the CPC's absorber which is acting as a saline water basin in a tubular arrangement. This specially designed CPC-CTSS required only 3 minutes warming up, as opposed to a typical warm-up period of an hour or more for basin type stills with no reflectors. The inner tube was cooled by flowing air at the rate of 4.5 m/s.

Design of the CPC

The reflector profile for the CPC [16] with a 'V' groove at the bottom is such that all rays entering the cavity opening end up at the absorber. It can be drawn for a tubular absorber of radius r_1 , and half acceptance angle θ_A allowing a small space δ between cavity opening and the absorber see Figure 1.

$$x = r_1 \sin\theta - \rho \cos\theta \tag{1}$$

The upper portion of the reflector profile AB and CD can be generated using the relation,

$$x = -r_1 \cos\theta - \rho \sin\theta \tag{2}$$

Where; ρ is the ground reflectance:

$$\rho = r_1 [\theta + \Delta \theta] \tag{3}$$

For the angles of

$$\pi - (2\varphi + \varepsilon) \le \theta \le \theta_A + \pi/2$$



Table 1 Still technical and operation details

S. no Climatic conditions Parameter		Parameter	Value		
1	Clear sky	Solar radiation (W/m ²)	652 - 1159		
		Ambient temperature (°C)	26.2 - 34		
		Relative humidity (%)	55 - 21		
		Average wind velocity (m/s)	1		
2	Design	Basin absorption (α_b)	0.96		
		Absorbtivity of cover (a_g)	0.05		
		Reflectance of cover	0.05		
		Transmittance of cover (τ_g)	0.9		
		Specific heat of water (C_w)	4190 J/(kg.K)		
		Length (m)	2		
		Width (m)	0.102		
		Radius of the receiver (m)	0.0158		
		Radius of the envelope (m)	2.2 x 10 ⁻² m		

$$\rho = \frac{r_1[\theta + \theta_A + \pi/2 + 2\Delta\theta - \cos(\theta - \theta_A)]}{[1 + \sin(\theta - \theta_A)]} \tag{4}$$

For the range of angles:

$$\pi/2 + \theta_A \leq \theta \leq 3\pi/2 - \theta_A$$

Where;

$$\Delta \theta = \cot \varphi - [\pi - (2\varphi + \varepsilon)] \tag{5}$$

And, from the Figure 1:

$$\varphi = \sin^{-1}(r_1/r_2) \tag{6}$$

$$\varphi + \varepsilon = \sin^{-1}[(r_1 + \delta)/r_2] \tag{7}$$

The upper portion of the radius of envelope is:

$$r_{2} = \frac{\left[r_{1}^{2} + (\delta/2\sin\alpha)^{2} + \delta r_{1}\right]^{1/2}}{\cos\alpha}$$
(8)

$$(1/\sin\alpha) - \sin\alpha = \delta/2r_1 \tag{9}$$

The bottom region of the profile can be modified by incorporating a V-shaped reflector portion just below the absorber.

The height 'h' and the open angle ' 2Ψ ' of the 'V' groove are related by the following relations:

$$\pi - 2\alpha \le 2\psi \le \pi/2 + \alpha \tag{10}$$

$$\sin \alpha = r_1 / (r + \delta + h) \tag{11}$$

Average fraction of radiation lost 'L' is given by:

$$L \approx \frac{\varepsilon^2}{2(1+2\varphi \tan \varphi)} \tag{12}$$



And the ratio of the concentration achieved to the maximum possible concentration of the concentrator is calculated as:

$$\frac{CR}{CR_{\max}} = \frac{\cot\phi + 2\phi + \varepsilon}{\pi}$$
(13)

Where;

$$CR_{\max} = 1/\sin\theta_A \tag{14}$$

Materials and methods

A 2 m concentric tubular solar still was designed and fabricated. The specification of the tubular solar still is



shown in Table 1. The inner and outer tubes were positioned with a 5 mm gap for the flowing air to cool the outer surface of the inner tube. An air blower was used to propel the air inside the tube at constant rate of 4.5 m/s. A rectangular basin of dimension 2 m \times 0.03 $\,m \times 0.025\,$ m was designed and coated with black paint using a spray technique. The surface was free of dust, dirt, rust and moisture before spraying. During operation, the water level in the basin decreased due to fast evaporation from the basin, so a dry spot appeared in the basin. This was avoided in successive trials by flowing the water continuously in the still with the help of a graduated tube. This continuous supply of water was maintained by a water storage tank which was kept near to the CPC still. The outlet of the storage tank was connected to the inlet of the CPC still. After saline water was poured inside the basin, the entire arrangement was sealed properly by using a rubber cork to ensure no air leakage. The cooling air had a constant flow rate of 4.5 m/s. Microprocessor digital anemometer (AM 4201) with anemometer vane probe was used to measure the flow rate of air inside the concentric tubes. The flow meter was calibrated for the actual flow rate at the operating temperature range. The process was repeated many times to assure a good accuracy and each time the flow rate were measured for a 1 minute interval.

The following parameters were measured every ten minutes: water temperature (T_w) , interior humid air temperature (T_a) , ambient temperature (T_{amb}) , total diffuse solar radiation (I), and distillate yield. The diffuse radiation is measured by a precision Pyranometer. All dimensions of the pictorial and schematic views of the experimental device are shown in Figures 2, 3 and 4.



Water quality analysis

The water quality analysis is performed at the Tamilnadu Agricultural University's, Soil Science and Agricultural Chemistry Department in Coimbatore, India. The results obtained are presented in Table 2. Two different water samples (A and B) are tested with their pH and electrical conductivity (in dS/m) measured before and after desalination. Before desalination, the level of electrical conductivity in the water of about 1 dS/m which is ~2% of ocean water, but not drinkable was obtained. However, after desalination it decreased to 0.10 dS/m⁻, which is drinkable. The typical pH varies from one water sample to another as well as on the nature of the construction

materials used in the water distribution system. It is usually in the range of 6.5 to 8.

Findings

Ahmed [17] has reported a continuous increase in output from a solar still with the increase in total solar radiation. Hourly variation of daily distillate output depends upon how the radiation is distributed throughout the day. The solar radiation and ambient temperature for a typical day are shown in Figure 5. The variation of ambient temperature was in the range of 24.7° C to 36.2° C and solar radiation received during the study was in the range of 362 W/m^2 to 1038 W/m^2 . Figure 6 shows the time

Table 2 Tested water g	uality re	sults
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Sample no.	TDS (mg/L)		рН		Conductivity (Ds/m)	
	Before desalination	After desalination	Before desalination	After desalination	Before desalination	After desalination
Sample A-B	320	40	7.60	7.32	1	0.10



variation of temperature with air flow over the inner tube. The water temperature was in the range of 45.7°C to 89°C without air flow and 34°C to 79.1°C with air flow. Figure 7 shows the variation of distilled yield with respect to time. The water collection rate obtained was 1428 mL/day without flow of air and 2020 mL/day with air flow. A significant increase in distilled water production was observed in the concentric tubular solar still with air flow over the inner tube. The average daily solar direct and diffuse incident radiation without and with air flow were

7080 Wh/m² and 6960 Wh/m², respectively. The total aperture was 2.04 m². With a heat of vaporization at boiling of 2260 J/g, the overall solar efficiencies without and with air flow were 16.2% and 18.9%, respectively.

Discussion

The driving force of the solar distillation technique is the difference between temperature of water in the still basin and the temperature of the cover (Tw-Tc) Arunkumar *et al.*, [6]. The existence of such temperature difference







ensures the continuation of the distillation process. Clearly, the tubular solar still with top cover air cooling is superior to the solar still without cooling. This increase of output is shown in Figure 7. The productivity of the hemispherical solar still is moderately improved by cooling the top cover. The variation of distilled yield is in the range of 17 ml to 113 ml for without cooling and 32 ml to 132 ml with

cooling. The daily average water productivity is increased from 1500 ml to 2200 ml for a fixed flow rate of 4.5 m/s of cold air fed. Figure 8 depicts the diurnal variation of efficiency with respect to time. The overall daily efficiency is calculated with the total radiation and total water output. The conventional still achieved 16.2%, and the cooling air system improved it to 18.9%. This implies that



the efficiency of the tubular solar still was improved by a factor of 1.25 due to the effect of air cooling. However, the advantage of the CPC is that the expensive still area is minimized and replaced with relatively inexpensive reflector. Therefore, the overall cost of distilled water should be lower.

The inlet and outlet temperatures [18] of the cooling air are shown in Figure 8. The extracted heat from the CPC-CTSS would allow further production of distilled water in a single slope solar still. With a 25°C rise and 4.5 m/s flow for 8 hours, the amount of heat was 140 Wh. With a 50% efficient single slope still, this could produce an additional 110 ml of distilled water, roughly 5% increase in output and efficiency. Air flow meter was kept near by the system to supply the cooling air. Its temperature was influenced by the ambient temperature and the solar radiation falling on it, which explains the variation in input temperature.

Conclusion

This research article summarized a new compound parabolic concentric-tubular solar still (CPC-CTSS), which has been designed for and tested under the climatic conditions of Coimbatore, India. The effect of cooling air flowing over the condensation surface was studied. The daily yield of CPC-CTSS was found 1445 mL/day and 16.2% efficiency without air flow and 2020 mL/day and 18.9% efficiency with air flow at a constant flow rate of 4.5 m/s. To bound in a nutshell, this innovative approach of concentrator assisted tubular solar still with air flow augments the performance with enhanced rate of evaporation and condensation with safer operation procedures.

Competing interests

The authors declare that they have no competing interests.

Authors' contributions

Arunkumar Thirugnanasambantham carried out the experiments under the guidance of Jayaprakash Rajan. All authors read and approved the final manuscript.

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Author details

¹Department of Physics, Dr. NGP Institute of Technology, Coimbatore 641 048 Tamilnadu, India. ²Department of Physics, Sri Ramakrishna Mission Vidyalaya College of Arts and Science, Coimbatore 41 020, Tamilnadu, India. ³Research and Development (Renewable Energy), NSP Green Energy Technologies Pvt. Ltd. Chennai, Tamilnadu, India.

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