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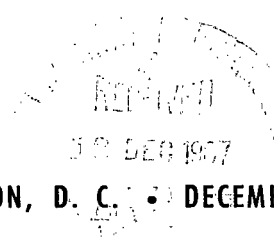NASA TN D-4286

EFFECT OF GRAVITY ON SELF-PRESSURIZATION OF SPHERICAL LIQUID-HYDROGEN TANKAGE

by John C. Aydelott

Lewis Research Center
Cleveland, Ohio

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SUMMARY

The information obtained from several experimental programs was examined in order to understand better the thermodynamic history of spherical, 9-inch- (23-cm-) diameter liquid-hydrogen tankage at both normal and reduced gravities. The rate of pressure rise in the hydrogen container was lower under reduced-gravity conditions than under normal-gravity conditions because of the increase in the liquid-wetted wall area and the increased boiling. The location of the sources of heat relative to the hydrogen liquid and vapor was the most important factor in determining the rate of pressure rise.

INTRODUCTION

The space exploration program of the NASA is heavily dependent on the use of liquid hydrogen as a rocket fuel. Hydrogen is very attractive for use as a rocket fuel because of the high specific impulse it produces either when reacted with an oxidizer or when used as a heated expellant as in a nuclear rocket.

Hydrogen has many properties which set it apart from common liquids. Because of its low density, it is practical, for space missions, to store hydrogen only as a liquid. The low equilibrium temperature of liquid hydrogen at atmospheric pressure makes it very difficult to insulate a storage container sufficiently to prevent a net heat gain. Under these conditions, the pressure will eventually rise until venting is required, with an accompanying propellant loss. Also of particular interest is the low thermal conductivity of hydrogen, which makes it possible for subcooled liquid and highly superheated vapor to coexist in the same container. This situation occurs when a closed system is exposed to heat, which causes an increase in the system pressure. The interface between the liquid and vapor phases remains at the saturation temperature corresponding to the increasing total system pressure. The average liquid temperature may increase at a slower rate, and thus the liquid bulk becomes subcooled; the average vapor temperature may increase

at a faster rate, and thus the vapor becomes superheated. As a consequence, simple thermodynamic analysis often can not predict the rate of pressure rise in a closed system containing liquid hydrogen.

An excellent review of the work in this field can be found in reference 1. The majority of the work has been restricted to cylindrical tanks with heating only on the side walls. Therefore, natural convection theory for vertical plates could be used to predict the heat and mass transfer within the liquid phase. In general, direct heating of the vapor was not considered. In this investigation, both the geometry of the hydrogen container and the basic heat-transfer mechanisms were different. Consequently, no attempt was made to correlate the resulting data with existing analyses, and no detailed discussion of other work is included.

Many variables affect the rate of pressure rise in a closed cryogenic container. The most important are container geometry and size, heat-transfer rate and distribution, percent liquid filling, and gravity level as it affects both the liquid-vapor configuration and the heat-transfer mechanisms involved.

For the past several years, the Lewis Research Center has been engaged in a program of zero-gravity research. Part of this research effort was conducted in the 2.2-second drop tower research facility to determine the static and dynamic behavior of both cryogenic and noncryogenic fluids under reduced-gravity conditions (refs. 2 to 4). Simultaneously, a series of experiments was conducted using Aerobee sounding rockets and an Atlas Scientific Passenger Pod. These studies were conducted to determine the dynamic behavior and thermodynamic history (pressure and temperature as a function of time) of liquid hydrogen under reduced-gravity conditions when contained in a closed 9-inch- (23-cm-) diameter sphere (refs. 5 to 9). A series of self-pressurization tests was conducted by the author to determine the thermodynamic history of a 9-inch- (23-cm-) diameter sphere under normal-gravity conditions (ref. 10). Various combinations of the variables percent liquid filling, heat-transfer rate, and heat-transfer distribution were studied.

This report presents a review of the research performed at Lewis, at both normal and reduced gravities, as it applies to the prediction of the thermodynamic history of spherical, 9-inch- (23-cm-) diameter, liquid-hydrogen tankage.

ANALYSIS

Theoretical Pressure Rise

The first law of thermodynamics is

$$Q = \Delta U + \int P \, dV$$

where

Q   total energy input to system under consideration

U   total system internal energy

P   absolute pressure

V   total system volume

For a closed, constant-volume system, all the heat absorbed by the system manifests itself in a change in the total internal energy of the system (i.e., for $dV = 0$, $Q = \Delta U$). If the system is a tank containing a liquid and its vapor, a knowledge of how the added heat affects the internal energy distribution and, thus, the temperature distribution within the tank makes possible prediction of the total system pressure. For a two-phase mixture, temperature and pressure are dependent variables at the interface between the liquid and the vapor.

The temperature distribution in a cryogenic storage tank is highly complex and is affected by many variables. The most important of these are tank geometry and size, gravity level, percent filling, heat-transfer rate, and heat-transfer distribution.

This report presents two simple pressure-rise models for a nonventing hydrogen container. These models are not intended to describe the process that actually takes place but are intended to be a means of comparing one set of experimental data with another. The position that experimental data assume in relation to the theoretical models on a plot of pressure against heat added is a qualitative indication of how energy is being distributed within the hydrogen container.

The first model assumes homogeneous conditions (saturation temperature) throughout the hydrogen container and is a common calculation that is performed to compare data of this type. The second model assumes that all the energy absorbed by the hydrogen container goes into evaporating the liquid and maintaining the vapor at the saturation temperature corresponding to total system pressure. The liquid-phase temperature remains constant at the saturation temperature corresponding to the sealoff pressure. Appendix B of reference 10 contains the development of these theoretical models based on the first law of thermodynamics.

Figure 1 is a plot of tank pressure as a function of heat added for the two energy-distribution models. The plot is for a 1-cubic-foot- (0.02832-cu-m-) container with initial liquid fillings of 25, 50, and 75 percent by volume. The reader may approximate the energy input, as determined by these models, that will cause a specified change in pressure for any tank size or filling. This approximation is possible by interpolating to determine the effect of the percent filling and by multiplying the heat added by the volume of the tank in cubic feet since the energy input is a linear function of the tank volume.
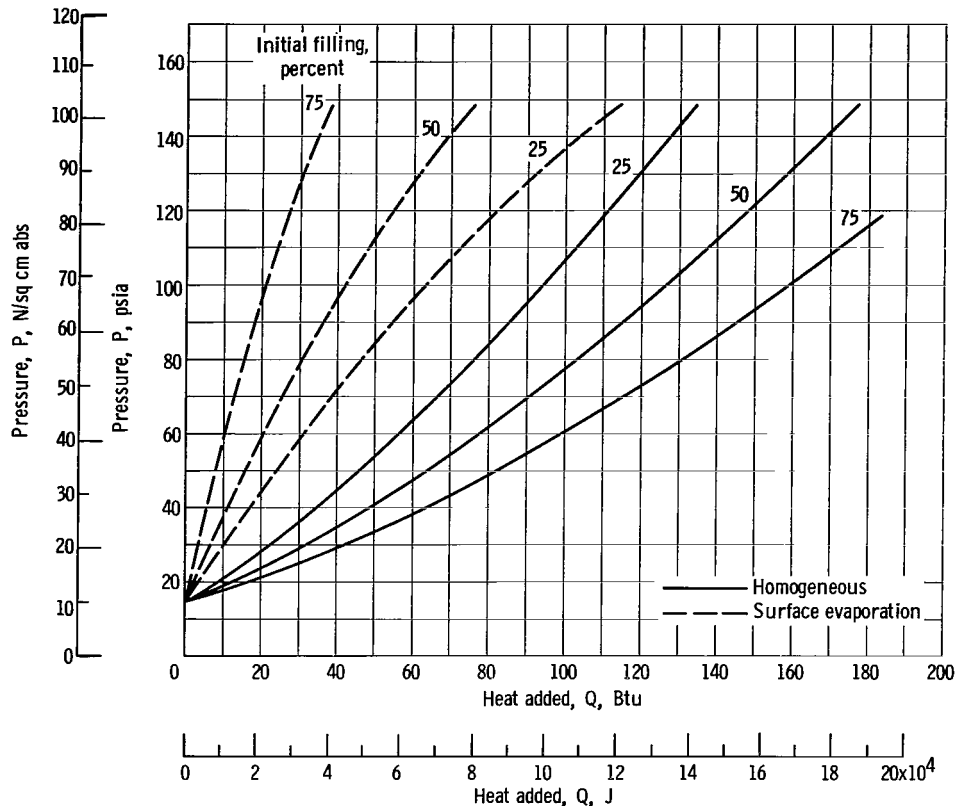
Figure 1. - Tank pressure as function of heat added for two energy-distribution models; 1-cubic-foot (0.02832 cu m) tank.

Heat Transfer

For each of the liquid-hydrogen pressurization experiments to be discussed, a heat-transfer analysis was performed so that the experimental data could be compared with the theoretical models. Each heat-transfer analysis makes it possible to determine the total energy input to each hydrogen container at any time. The reader is referred to the corresponding individual reports (refs. 5 to 10) for the details of each analysis. The experimental results are presented as plots of total tank pressure as a function of total heat added. The two theoretical models for the corresponding conditions are included for comparison.

ADIABATIC CONFIGURATION EXPERIMENTAL STUDIES

The experimental program to determine the effect of weightlessness, with no heat addition, on the liquid-vapor configuration for both cryogenic and noncryogenic fluids

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contained in spherical tanks was conducted in the Lewis 2.2-second drop tower. More detailed information about the research facility and the experiment packages can be found in references 2 and 3.

The fluids studied have static zero-degree contact angles with glass. In zero gravity, the liquid-vapor interface configuration for these fluids is a spherical vapor bubble that has no preferred location in the interior of the liquid. This configuration results from the attempt of the liquid-vapor interface to maintain its characteristic contact angle with the glass wall; this could not be accomplished because of the continuous curvature of the wall. The resulting spherical vapor bubble is the equilibrium configuration resulting from a minimization of the surface energy.

LIQUID-HYDROGEN PRESSURIZATION EXPERIMENTAL STUDIES

Apparatus and Procedure

Simultaneously with the drop tower studies, two programs were undertaken to determine the effects of reduced gravity and heat addition on the rate of pressure rise in
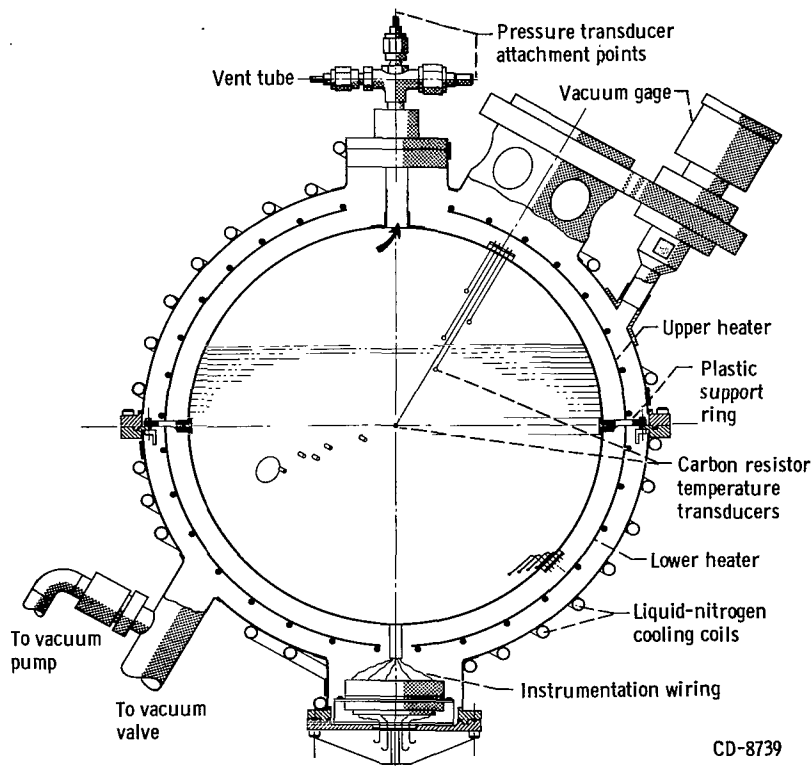
Figure 2. – Apparatus for typical Aerobee experiment.

9-inch- (23-cm-) diameter spherical liquid-hydrogen tanks. For one of the programs, Aerobee sounding rockets were used to obtain approximately 5 minutes of time in a reduced-gravity environment for a series of experiments. A good description of the Aerobee rocket and its use in reduced-gravity research is contained in reference 6. The second program (ref. 9) used an Atlas Scientific Passenger Pod to obtain 21 minutes of reduced-gravity environment.

The experimental apparatus for both programs were similar in that they consisted of three concentric spheres; the inner sphere contained the liquid hydrogen, the intermediate sphere had electric heating coils mounted on the exterior surface, and the outer sphere served as a vacuum jacket. The primary energy input to the inner sphere was radiant exchange from the intermediate sphere. A typical experiment designed to be carried by the Aerobee rocket is shown in figure 2. The experiment for the Atlas pod is shown in figure 3. Each experiment was designed to explore the effect of a different combination of the variables percent filling, heat-transfer distribution, and heat-transfer rate. The difference between each of the Aerobee experiments and the Atlas pod experiment was primarily limited to instrumentation.

More recently a series of tests was performed under normal-gravity conditions, using liquid hydrogen contained in a modified Aerobee experimental apparatus (ref. 10).
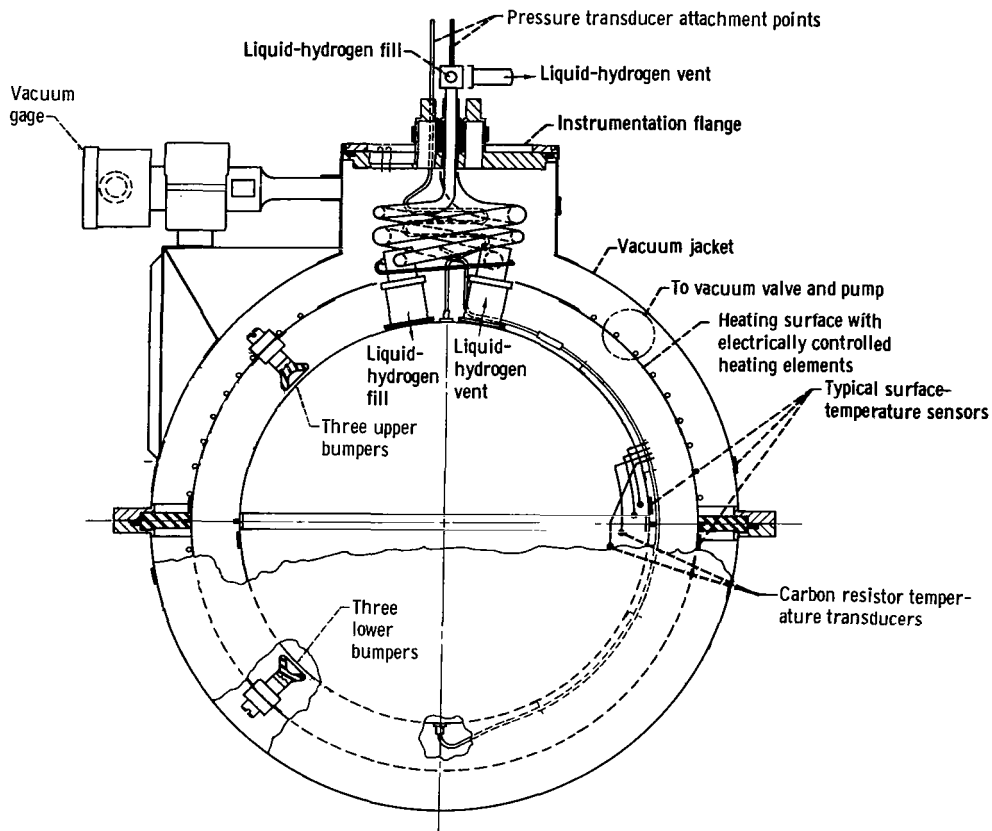

Figure 3. - Apparatus for Atlas Scientific Passenger Pod experiment.

In addition to having instrumentation designed to meet the needs of normal-gravity testing, the intermediate sphere was designed so that either hemisphere could be removed to change the heating configuration. The hydrogen container was subjected to various combinations of the variables percent filling, heat-transfer rate, and heat-transfer distribution (top, bottom, or uniform heating).

For the Aerobee and the Atlas pod programs, the hydrogen container was sealed at lift-off and allowed to self-pressurize for the duration of the flight. For the normal-gravity tests, the experiment was sealed at a convenient time and allowed to self-pressurize to a maximum pressure of 100 psia (68.95 N/sq cm abs). The inner sphere pressure, the vacuum-jacket pressure, and the surface temperatures of the three concentric spheres were measured in all the studies. In addition, the experiments of references 7 to 10 had carbon resistor temperature transducers inside the inner sphere to determine the temperature of the hydrogen liquid and vapor. A detailed discussion of the instrumentation used on each experiment, instrumentation calibration, data recording, chill-down, filling and operating procedure, is included in references 5 to 10. An analysis of the error associated with the temperature and pressure transducers can be found in reference 10.
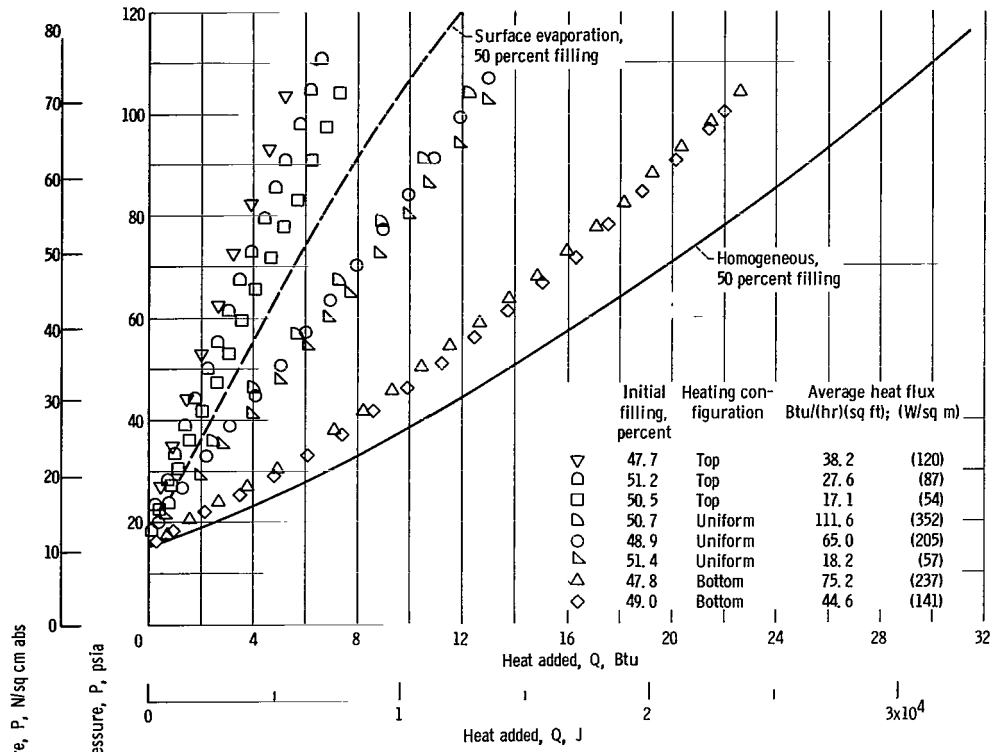
Normal-Gravity Test Results

Twenty-one quiescent tests were performed under normal-gravity conditions (ref. 10). Average heat fluxes ranged from 15.5 to 128.6 Btu per hour per square foot (49 to 405 W/sq m) and the percent liquid filling ranged from 29.3 to 80.4 percent by volume. The effects of uniform heating, bottom heating, and top heating were explored.

Effect of heat-transfer rate and distribution. - Figure 4(a) shows the effects of heat-transfer rate and distribution on the pressure as a function of heat added. The tank was approximately 50 percent full for these quiescent tests. The 50-percent filling was chosen since it represents the least complicated geometric situation, where the liquid-vapor interface and the division between the upper and lower heaters are approximately in the same horizontal plane.

Figure 4(a) shows that the heating configuration has a much greater effect than the heat-transfer rate on the slope of the pressure against heat added data. The heat-transfer rate had the least effect on the slope for the bottom-heating tests. This influence increased for the uniform-heating and top-heating tests. However, the heat-transfer rate was definitely secondary in importance to the heating configuration.

The two coordinates, pressure and heat added, as plotted in figure 4(a), are the integrals over time of the pressure-rise rate and the heat-transfer rate. Coincident test data on this plot would indicate a linear relation between the pressure-rise rate and the

Pressure, P, N/sq cm abs

Pressure, P, psia

120

80

70

100

60

80

50

40

60

30

40

20

20

10

0   0

Surface evaporation,
50 percent filling

Homogeneous,
50 percent filling

	Initial con-filling, percent	Heating con-figuration	Average heat flux Btu/(hr)(sq ft);	(W/sq m)
▽	47.7	Top	38.2	(120)
△	51.2	Top	27.6	(87)
□	50.5	Top	17.1	(54)
◁	50.7	Uniform	111.6	(352)
○	48.9	Uniform	65.0	(205)
▷	51.4	Uniform	18.2	(57)
△	47.8	Bottom	75.2	(237)
◇	49.0	Bottom	44.6	(141)

0   4   8   12   16   20   24   28   32

Heat added, Q, Btu

0   1   2   3x10⁴

Heat added, Q, J

(a) Effect of heat-transfer rate and distribution.

120

80

70

100

60

80

50

40

60

30

40

20

20

10

0   0

Surface
evaporation;
filling,
percent

76.5

48.9

34.9

Homogeneous;
filling,
percent

34.9

48.9

76.5

	Initial filling, percent	Average heat flux, Btu/(hr)(sq ft);	(W/sq m)
◇	76.5	72.6	(229)
○	48.9	65.0	(205)
△	34.9	59.9	(189)

0   4   8   12   16   20   24   28

Heat added, Q, Btu

0   1   2   3x10⁴

Heat added, Q, J

(b) Effect of percent filling.

Figure 4. - Pressure as function of total heat added for normal-gravity quiescent tests.

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heat-transfer rate; that is, doubling the heat-transfer rate doubles the pressure-rise rate. For the bottom-heating tests this linear relation was followed almost exactly, but the uniform-heating tests and, to a greater degree, the tests with top heating only began to deviate. This difference indicates that the energy distribution in the liquid was unaffected by the rate of energy input. However, the rate of energy input to the vapor greatly affected the temperature or energy distribution.

Reference 10 concludes that the mode of heat transfer in the liquid in a spherical tank is either turbulent convective currents throughout the bulk or boiling. An essentially uniform temperature would be anticipated in the liquid bulk for either turbulent-convection or boiling heat transfer. Because a uniform temperature was experimentally observed in the liquid bulk, it was concluded that at the lower heat fluxes the heat-transfer mechanism was dominated by turbulent convection. At the higher heat fluxes, some boiling was encountered. The heat-transfer processes which take place in the vapor are not clearly understood, but the primary mode of energy transfer may be intermolecular (i.e., conduction and diffusion).

The theoretical surface-evaporation line (saturated vapor, no liquid bulk heating) and the homogeneous line (uniform-temperature liquid and vapor) are plotted in figure 4(a) to permit comparison of the experimental data with these models. The top-heating tests approach the surface-evaporation model in one respect; the liquid is heated a very slight amount, but superheating the vapor pushes the experimental data above the theoretical surface-evaporation line. The bottom-heating tests approach the homogeneous model in one respect; the liquid is nearly saturated, but some heating of the vapor causes superheating, and the experimental data lie above the theoretical homogeneous line. The uniform-heating tests combine some heating of the liquid with superheating of the gas; the resulting data lie between the two extremes of top heating and bottom heating.

Effect of percent filling. - Figure 4(b) shows the effect of the percent filling on the pressure as a function of heat added for three uniform-heating tests. These three tests were chosen to demonstrate the effect of the percent filling because the average heat flux was nearly the same. Here again, the experimental data are compared with the theoretical models. Based on figure 4(a), if the average heat flux had been the same, the experimental data would have been somewhat closer together than those shown in figure 4(b). The conclusion to be drawn from figure 4(b) is that the rate of pressure rise was only slightly affected by varying the percent filling when the heating was uniform. However, a trend toward higher rates of pressure rise resulted from higher fillings.

The top-heating tests (ref. 10) exhibited increasing rates of pressure rise with increased filling, similar to the theoretical surface-evaporation model. This increase in pressure is a result of the increasing unheated liquid mass and the decreasing vapor volume which must absorb the incoming energy. The bottom-heating tests exhibited decreasing rates of pressure rise with increased filling, similar to those for the theoretical homogeneous model. This decrease in pressure results from the increased, nearly
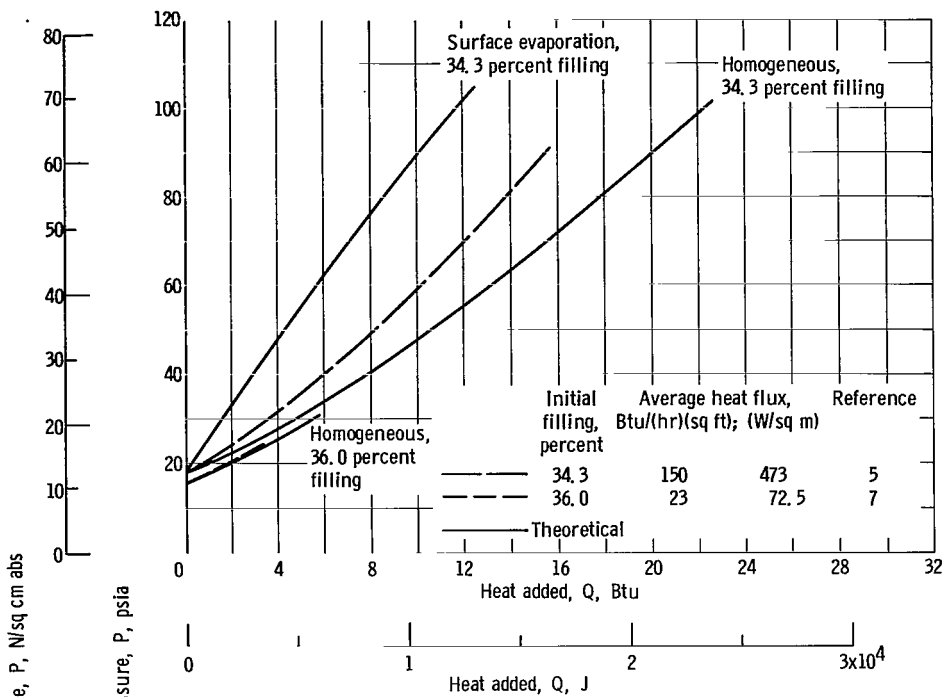
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saturated, liquid mass which is available to absorb the incoming energy. Evidently, the uniform-heating tests are slightly dominated by the heating of the vapor, which causes a small increase in the rate of pressure rise with increased filling.
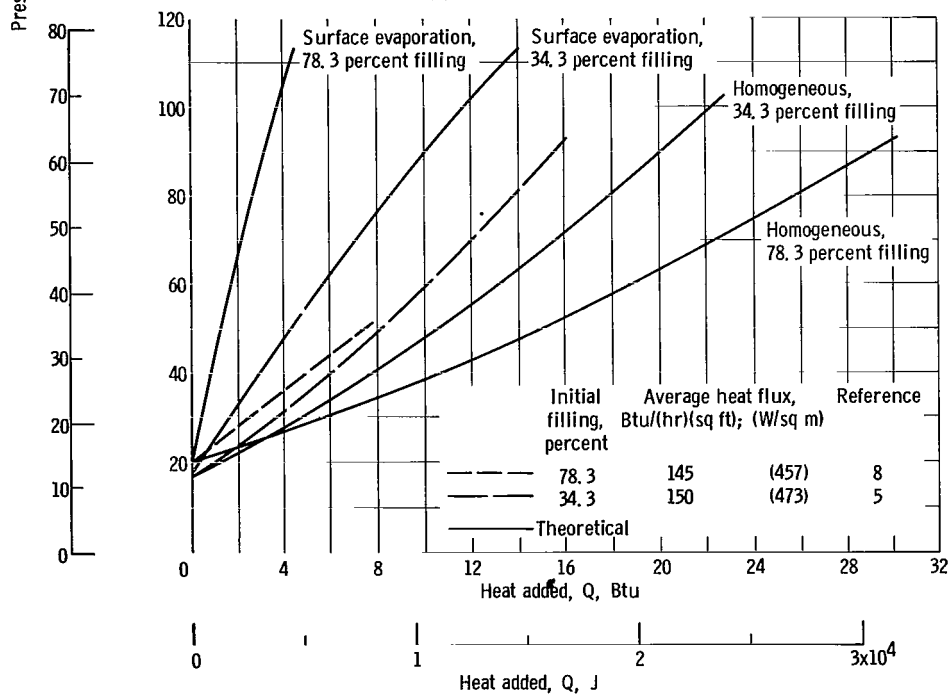
Reduced-Gravity Test Results

Effect of heat-transfer rate and distribution. - The effect of heat-transfer rate on the thermodynamic history of two reduced-gravity experiments is shown in figure 5(a). Both of the experiments were flown on Aerobee sounding rockets, so that approximately 5 minutes of test time was available. The data of reference 5 are from an experiment with a uniform heat flux of 150 Btu per hour per square foot (473 W/sq m) and an initial liquid filling of 34.3 percent by volume. An acceleration field of less than 2×10^{-3} g was experienced while the rocket and the experiment were in free fall. The data of reference 7 are from an experiment with a uniform heat flux of 23 Btu per hour per square foot (72.5 W/ sq m) and an initial liquid filling of 36.0 percent. The corresponding free-fall acceleration was less than 4×10^{-4} g. The random gravity forces experienced by both experiments were small enough so that the liquid hydrogen was in the capillary dominated region.

In the case of reference 7, there was less total heat added during approximately the same test time because of the lower heat-transfer rate. As might have been expected from the adiabatic studies, the liquid hydrogen in the experiment with the lower heat-transfer rate quickly wetted the tank walls following entry into the low-gravity environment. The resulting vapor bubble was nearly centrally located, although it may have been held in that position by the carbon resistor temperature transducers. The resistors were designed to measure the liquid temperature but could also have acted as small surface-tension baffles. As a result, all the incoming energy passed through the liquid-wetted walls and produced a nearly homogeneous mixture. This situation is similar to the normal-gravity bottom-heating tests, where only the liquid was heated and nearly homogeneous conditions were observed.

The experiment at a higher heat-transfer rate (ref. 5) also resulted in complete wetting of the tank walls upon entering reduced gravity. However, the vapor bubble drifted to one end of the tank, and shortly thereafter a dry spot appeared. The dry area was initially where the fill and vent tube is attached, the area of the highest localized heat-transfer rate. As time progressed, the dry area continued to grow until only one-half of the inner sphere was wet. This spreading of the dry area results from the localized energy input to the inner sphere evaporating the liquid hydrogen faster than capillary flow can replace it. The resulting liquid-vapor interface then begins to approach a normal-gravity configuration. The direct heating of the vapor causes the rate of pressure rise to be greater than homogeneous, as in the uniform-heating normal-gravity tests.

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Pressure, P, N/sq cm abs

Pressure, P, psia

Surface evaporation, 34.3 percent filling

Homogeneous, 34.3 percent filling

Homogeneous, 36.0 percent filling

Initial filling, percent	Average heat flux, Btu/(hr)(sq ft); (W/sq m)		Reference
34.3	150	473	5
36.0	23	72.5	7
Theoretical			

Heat added, Q, Btu

Heat added, Q, J

3x10⁴

(a) Effect of heat-transfer rate.

Surface evaporation, 78.3 percent filling

Surface evaporation, 34.3 percent filling

Homogeneous, 34.3 percent filling

Homogeneous, 78.3 percent filling

Initial filling, percent	Average heat flux, Btu/(hr)(sq ft); (W/sq m)		Reference
78.3	145	(457)	8
34.3	150	(473)	5
Theoretical			

Heat added, Q, Btu

Heat added, Q, J

3x10⁴

(b) Effect of percent filling.

Figure 5. - Pressure as function of total heat added for reduced-gravity tests. Uniform heating configuration.

(c) Effect of disturbances.

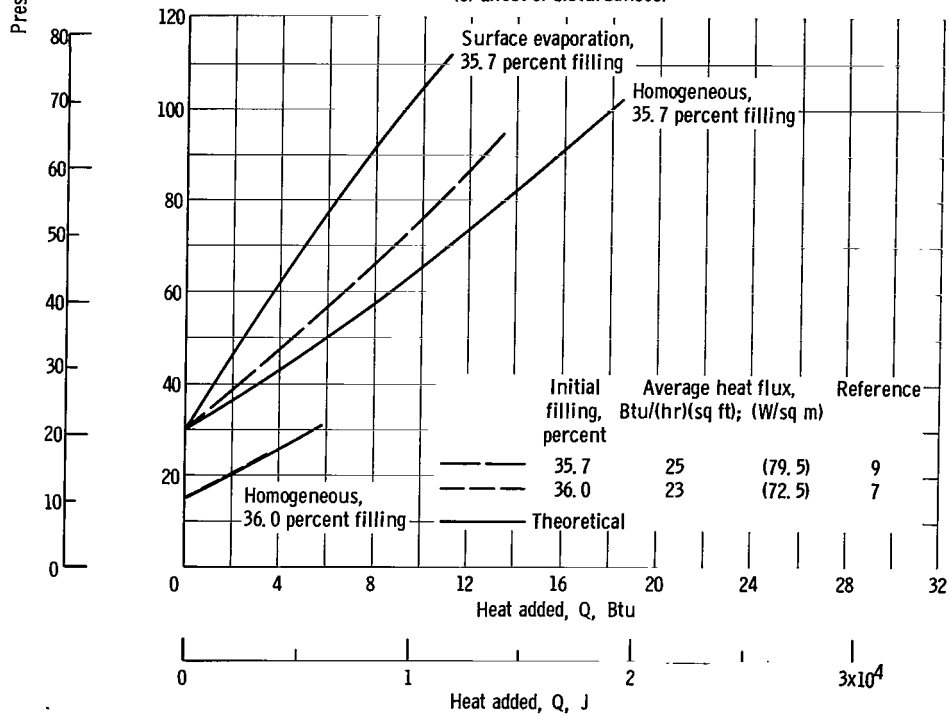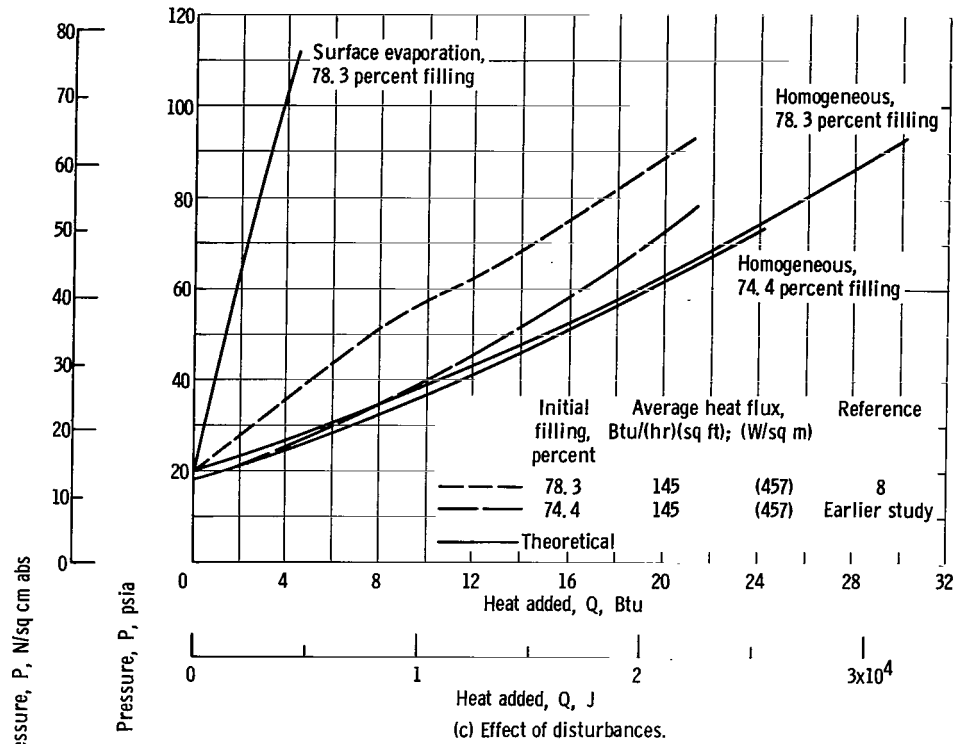(d) Effect of small unidirectional acceleration.

Figure 5. - Concluded.

This effect was also reported in reference 6 which presents the data for a low percent filling, unsymmetrically heated experiment. The average rate of energy input was approximately the same as that reported in reference 7 (23 Btu/(hr)(sq ft); 72.5 W/sq m). The vapor bubble moved to the area of maximum heat addition and drying occurred. The experimental results are nearly identical to those of reference 5 shown in figure 5(a). The unsymmetrical heating caused nearly the same degree of nonhomogeneity as symmetrical heating (ref. 5), but at a lower heat-transfer rate.

Effect of percent filling. - The effect of liquid filling on the thermodynamic history of two reduced-gravity experiments is shown in figure 5(b). The data obtained from reference 8 are compared with the data from reference 5. The data of reference 8 are for an experiment with a uniform heat flux of 145 Btu per hour per square foot (457 W/sq m) and an initial liquid filling of 78.3 percent by volume. The acceleration field was nominally 10^{-3} g for the first portion of the flight, which is shown in figure 5(b). The latter portion of the flight is discussed in connection with figure 5(c). The data of reference 5 were for approximately the same heat flux and gravity level, but for a lower liquid filling of 34.3 percent.

The rate of pressure rise was slightly less for the test with a higher percent filling. The container walls in the high-filling experiment were completely wetted for the entire reduced-gravity period. The vapor bubble was near the top of the container during the first part of the flight, but liquid separated the vapor from the tank wall. Consequently, all the energy entered the hydrogen container through liquid-wetted walls. For this high-filling test, the internal instrumentation may have acted as a surface-tension baffle, as previously discussed in connection with reference 7. These carbon resistor temperature transducers measured the liquid temperature and were located in a pattern of concentric spheres.

The heat fluxes which were employed in the flight experiments are above the normal-gravity incipient boiling point. In reduced gravity, convective heat transfer diminishes so that nucleate boiling is the primary mode of energy exchange. For the low-filling and low-heat-transfer-rate case (ref. 7, fig. 5(a)), the resulting energy distribution was nearly homogeneous. For the high-filling and high-heat-transfer-rate case (ref. 8), the resulting energy distribution was definitely not homogeneous even though both experiments had completely wetted walls. A possible explanation of these results is that bubbles formed on the wall of the tank and did not penetrate very far into the liquid bulk. The bubbles may condense close to the wall and thus heat only a small portion of the liquid volume. Also, the lower heat-transfer rate associated with reference 7 makes it possible for more energy to be conducted into the liquid away from the wall. These conclusions are supported by the liquid temperature gradients, which were much greater for the high-filling test than for the low-filling test. These large temperature gradients then explain the nonhomogeneous pressure rise shown in figure 5(b), since a homogeneous pressure rise results from a uniform-temperature system.

Effect of disturbances. - As the flight of reference 8 continued, the gravity level decreased to a final value of approximately 2×10^{-4} g. At about the middle of the test, the vapor bubble moved because of this reduction in gravity level and possibly because of the surface-tension effect of the internal instrumentation. This motion of the bubble caused a general mixing of the liquid in the tank, and a reduction in the rate of pressure rise resulted, as shown in figure 5(c).

Also shown in figure 5(c) are some previously unpublished data from an earlier test. Because the Aerobee rocket is spin stabilized during the boost phase, it is necessary to keep the experiment from rotating by using a despin table. Following the boost phase, a small nozzle system is used to despin the rocket casing so that no disturbances due to misalinement of the rocket and experiment centers of gravity will be transmitted to the experiment during the coast phase of the flight. This despin system failed on the first attempt to determine the effect of higher fillings on the thermodynamic history of a spherical hydrogen container. The resulting accelerations of the experiment were $\pm6\times10^{-2}$ g and caused a mixing action to take place in the hydrogen container. The resulting rate of pressure rise was lower than that of reference 8. These experiments were identical, but the despin system operated satisfactorily in the case of reference 8. The conclusion to be drawn from figure 5(c) is that random disturbances cause a general mixing of the liquid hydrogen and thus a more nearly homogeneous condition.

Effect of small unidirectional acceleration. - In contrast to the previous conclusion is the effect of a small unidirectional acceleration presented in figure 5(d). The data from reference 7 are again shown and compared with the results of the Atlas Scientific Passenger Pod experiment (ref. 9).

The Atlas Pod experiment was performed under the same test conditions as reference 7 with the exception of having available a longer period of reduced gravity. The heat-transfer rates, the percent liquid filling, and the internal instrumentation were nearly identical. When the pod was separated from the launch vehicle, the ejection system malfunctioned and caused the pod to tumble. The resulting centrifugal acceleration, due to the location of the experiment relative to the center of gravity of the pod, was calculated to be a constant 10^{-3} g acting in the same direction as a normal gravity field. This low-gravity field moved the liquid to the bottom of the tank and the vapor to the top.

After approximately 1 minute in this 10^{-3}-g field, the inner sphere began to dry in the region of the fill and vent tubes. As time progressed, this dry area increased to approximately 25 percent of the inner sphere. Exposing this increasing area of vapor to the incoming energy slowly increased the rate of pressure rise. This result is similar to that obtained in reference 5, where a higher heat-transfer rate was employed.

Comparison of Normal- and Reduced-Gravity Test Results

Figure 6 is a plot of pressure as a function of heat added for the reduced-gravity tests reported in references 5 and 8 and for two normal-gravity tests of reference 10. All these tests were performed at relatively high heat-transfer rates. For the high-filling tests, the rate of pressure rise is greater for the normal-gravity test than for the reduced-gravity test. For the low-filling tests, the rate of pressure rise is initially greater for the normal-gravity test than for the reduced-gravity test. With increasing time, the reduced-gravity, low-filling experiment experiences wall drying so that eventually the slopes of the normal-gravity and the reduced-gravity test data are approximately the same.

In general, the liquid-wetted wall area increased with decreasing gravity level. Also, as the gravity level is reduced, there is a transition from convective heat transfer to
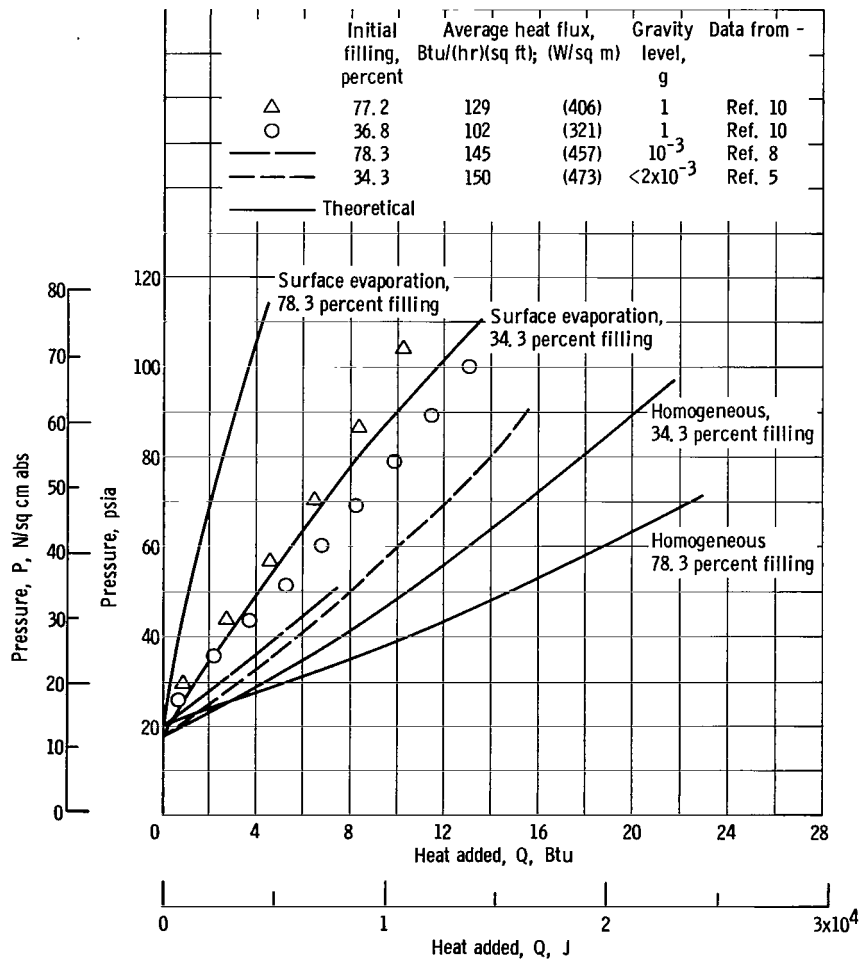

Figure 6. – Pressure as function of heat added; comparison of normal-gravity and reduced-gravity tests. Uniform heating configuration.

more active boiling, which mixes the liquid bulk. Both the decreasing dry wall area and the increased boiling reduced the temperature gradients in the hydrogen and thus lowered the rate of pressure rise.

SUMMARY OF RESULTS

The information obtained from several experimental programs was examined in order to understand better the thermodynamic history of spherical, 9-inch- (23-cm-) diameter liquid-hydrogen tankage. These programs included a series of Aerobee Sounding Rocket experiments, an Atlas Scientific Passenger Pod experiment, and data obtained from the Lewis drop tower. This review yielded the following results.

1. Under zero-gravity adiabatic conditions, a zero-degree-contact-angle liquid will completely cover the walls of a spherical tank. The resulting spherical vapor bubble has no preferred location within the interior of the liquid.

2. Under normal-gravity nonadiabatic conditions

a. The rate of pressure rise was affected most by the location of the heat addition relative to the liquid location, being greatest for top heating and least for bottom heating.

b. The rate of pressure rise increased almost linearly with increasing heat-transfer rate.

c. For the uniform-heating tests, the rate of pressure rise was only slightly affected by varying the percent filling.

3. Under reduced-gravity uniform-heating conditions

a. The rate of pressure rise was greatly affected by the liquid configuration. Totally liquid-wetted walls produced the lowest rate of pressure rise.

b. The liquid configuration was dependent on the surface tension of the test fluid, on the heat-transfer rate and distribution, on the percent liquid filling, and possibly, in some cases, on the internal instrumentation. High heat-transfer rates and low percent liquid filling decreased the wetted wall area and increased the rate of pressure rise.

c. Oscillatory or random disturbances decreased the rate of pressure rise.

d. A small unidirectional acceleration tended to decrease the area of the liquid-wetted wall and to increase the rate of pressure rise.

4. In general, the rates of pressure rise under normal-gravity conditions were greater than those under reduced-gravity conditions for the following reasons:

a. The liquid-wetted wall area increased with decreasing gravity level.

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b. As the gravity level is reduced, convective heat transfer is diminished and boiling is enhanced. The increased bubble population acts to cause more complete mixing of the liquid bulk.

Lewis Research Center,
    National Aeronautics and Space Administration,
        Cleveland, Ohio, June 21, 1967,
            124-09-03-01-22.

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