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# FULL-SCALE TESTS OF A SHORT-LENGTH, DOUBLE-ANNULAR RAM-INDUCTION TURBOJET COMBUSTOR FOR SUPERSONIC FLIGHT

by Porter J. Perkins, Donald F. Schultz, and Jerrold D. Wear Lewis Research Center Cleveland, Obio 44135



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#### FULL-SCALE TESTS OF A SHORT-LENGTH, DOUBLE-ANNULAR

#### RAM-INDUCTION TURBOJET COMBUSTOR

#### FOR SUPERSONIC FLIGHT

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Lewis Research Center

#### SUMMARY

Full-scale tests were conducted to determine the performance of a short-length, 40-inch (102-cm) diameter turbojet combustor. The combustor was designed for a large turbofan engine operating at flight speeds up to Mach 3.0. The short overall diffusercombustor length of  $20\frac{1}{4}$  inches (51.5 cm) was achieved by using both the double-annulus and the ram-induction concepts. Single annular combustors for the same airflow are longer by 50 percent, or more, than the combustor described herein. The doubleannulus concept reduces combustor length while maintaining an adequate ratio of length to annulus height in each combustion zone. Ram induction reduces overall length by using more of the high velocity in the compressor discharge air. This shortens the diffuser by not diffusing to as high a static pressure and shortens the combustor by promoting rapid mixing in the primary and secondary zones.

Combustor performance was satisfactory in equalling or exceeding that of a similar ram-induction combustor that had a single annulus and was 50 percent longer. Combustion efficiency at design conditions was 100 percent, but at off-design conditions of low temperature and pressure was less than 100 percent.

The exit radial average temperature profile was in good agreement with the design profile. Exit temperature pattern factors measured 0. 20 and 0. 25 at cruise and takeoff conditions, respectively.

The overall diffuser-combustor total pressure loss was 6.2 percent and 8.4 percent for the takeoff and cruise conditions.

During 100 hours of operation at variable operating conditions, the durability appeared satisfactory except at the Mach 3.0 cruise condition  $(1150^{\circ} \text{ F} (895 \text{ K}) \text{ inlet temperature})$  where some burning occurred.

Combustor blowout limits and altitude relight were determined. The exit temperature response to a rapid step increase in fuel flow was also evaluated. Smoke measurements indicated no visible smoke at the cruise and takeoff conditions.

#### IN TRODUCTION

A shorter length combustor is desired in advanced jet engines on supersonic aircraft. The shorter length will reduce engine size and weight and improve combustor life. Supersonic speeds impose high inlet-air temperatures (about  $1000^{\circ}$  F) (811 K) on the combustor which requires large amounts of air to cool the metal surfaces. Less liner area in the shorter length can reduce the amount of inlet air required for cooling or increase the cooling capability thereby providing longer life. An increase in the amount of air available for dilution can also be provided. The Lewis Research Center is, therefore, engaged in research to develop short-length combustors.

The combustor development reported herein has double-annular and ram-induction concepts to achieve short length. The double annulus is used to provide the necessary length-to-height ratio but over a reduced length. The ram-induction principle is used to shorten the diffuser and to promote rapid mixing within a short length. The length of this combustor is compared with conventional can-annular and full-annular combustors in figure 1. For the same airflow, single annular combustors are about 65 percent longer and can-annular about 130 percent longer. The design operating requirements for the short combustor are the same as those for the combustor used in the Pratt & Whitney candidate supersonic transport engine primary combustor. The photograph in figure 2 shows the comparative length of the two combustors without exit transition liners which were the same length in both combustors.

Sector testing described in references 1 and 2 aided in the development of the double-annular combustor (sometimes referred to as the Twin Ram-Induction Combustor). The investigation reported herein, was conducted in a full-scale engine component, connected-duct research facility at Lewis. The combustor operating environment such as inlet-air temperatures, pressures, and airflows was simulated and detailed instrumentation, not possible in an engine installation, was used. Details of the test facility and instrumentation are contained in appendixes A and B.

The purpose of this investigation was to determine the performance characteristics of the full-scale double-annular combustor designed from the Pratt & Whitney sector studies (refs. 1 and 2). The fuel used for the tests was ASTM A-1 at ambient temperatures.

#### COMBUSTOR DESIGN

#### The Double-Annular Concept

The combustor used in this investigation is referred to as a double-annular, raminduction combustor. Constructing the combustion zone as a double annulus permits the reduction of overall combustor length while maintaining an adequate ratio of length to annulus height in each combustion zone. This feature allows a considerable reduction in length to be made over a single annulus with the same overall height.

Individual control of the inner and outer annulus fuel systems of the double-annular combustion zone provides a useful method for adjusting the outlet radial temperature profile.

#### The Ram-Induction Concept

The ram-induction combustor differs from the more conventional combustors in that the compressor discharge air is allowed to penetrate into the combustion and mixing zones without diffusing to as high a static pressure. The kinetic energy of the inlet air is thereby used to promote rapid mixing of air and fuel in the primary zone and diluent air and burned gases in the mixing zone. The airflow is efficiently turned into the combustor by two rows of vaned turning scoops that penetrate into the combustor.

The ram-induction combustor has several advantages over conventional static pressure-fed combustors. These are

(1) A shorter length combustor is obtained because more controlled mixing can be established in the combustion zone. This is achieved by better control of the airflow injection angle with the vaned turning scoops.

(2) Diffuser length can be shortened since diffusion to very low Mach numbers is no longer needed or desired. The overall diffuser-combustor length can, therefore, be reduced. The small area ratio diffuser in the shorter length could have less pressure loss since it is not as prone to flow separation. However, this advantage can be offset by the increased turning losses associated with spreading the relatively high velocity flow evenly around the combustor.

(3) The high velocity flow over the exterior surfaces of the combustor provides substantial convective cooling of these walls. This reduces the film cooling air requirements. Thus more air is available for mixing and temperature profile control.

A more detailed discussion of the ram-induction concept is provided in reference 3.

#### Combustor Design Details

The double-annular ram-induction combustor and associated diffuser design used for this investigation is shown in cross section in figure 3. Forward airflow spreaders in the diffuser split the inlet airflow into three passages leading into the combustor. These are the inside liner passage, the outside liner passage, and the center passage. The airflow around the outside and inside of the diffuser is ducted by shrouds surrounding the outside of both the outer and inner liners of the combustor. The high airflow velocity which is maintained from the diffuser inlet through this ducting is turned into the combustor burning zones by means of the scoops. The first row of scoops supplies air to the primary zone while the second row supplies diluent air to the secondary zone.

Basic dimensions for this combustor are shown in figure 3. The diameters are essentially those of the combustor for the Pratt & Whitney Aircraft experimental supersonic transport engine (JTF17 (ref. 4)). However, the diffuser-combustor overall length of the double-annular combustor is about 30 percent shorter than that used in the JTF17 engine.

Photographs of the combustor are given in figure 4. Figure 4(a) shows the downstream end with the two circumferential rows of scoops attached to the inner and outer liners and to the center section. Figure 4(b) is a closeup of this same view showing more detail of the scoop arrangement. The fuel nozzles and associated swirlers are removed, but the deflectors for cooling the inner and outer headplates are shown at the nozzle locations. A side view of the combustor with the upstream diffuser airflow spreaders and inner exit transition liner added to the combustor is shown in figure 4(c). The notches in the airflow spreaders fit around the diffuser struts. The combustor is pin mounted through the struts using tangs attached to the inner and outer headplates that extend forward into the airflow spreaders.

<u>Fuel nozzles</u>. - Simplex fuel nozzles were used for the investigation. Two sets of nozzles were used to cover the range of high fuel flows for simulated takeoff conditions to low flows for altitude relight. Curves of total fuel flow for the combustor (sum of 64 nozzles) as a function of pressure drop across these nozzles are shown in figure 5.

<u>Combustor design specifications.</u> - The major items of the combustor design are tabulated in table I. The circumferential locations of combustor components such as scoops, fuel nozzles, and diffuser struts are shown in figure 6. The flow areas as distributed among the many openings (scoops, film cooling, swirlers, etc.) are given on the combustor sketch of figure 7. The scoop discharge areas with length and width dimensions are listed in table II.

#### CALCULATIONS

#### **Combustion Efficiency**

Efficiency was determined by dividing the measured temperature rise across the combustor by the theoretical temperature rise. The theoretical rise is calculated from the fuel-to-air ratio, fuel properties, inlet air temperature, and the amount of water

vapor present in the inlet airflow. The exit temperatures were measured with five-point traversing aspirated thermocouple probes and were mass-weighted for the efficiency calculation. The indicated readings of all thermocouples were taken as true values of the total temperatures. The mass-weighting procedure is given in reference 2. In each mass-weighted average, 585 individual exit temperatures were used.

#### Reference Velocity and Diffuser Inlet Mach Number

Reference velocity  $V_{ref}$  for the combustor was computed from the total airflow, the maximum cross-sectional area between the inner and outer shrouds (see table I), and the air density based on the total pressure and temperature at the diffuser inlet. Diffuser inlet Mach number was calculated from the total airflow, the total temperature, static pressure measured at the diffuser inlet, and the inlet annulus area.

#### Total Pressure Loss

The total pressure loss  $\Delta P/P_{t3}$  was calculated by mass-averaging total pressures measured upstream of the diffuser inlet and at the combustor exit. The total pressure loss, therefore, includes the diffuser loss.

#### Exit Temperature Profile Parameters

Three parameters of interest in evaluating the quality of exit temperature profile are considered. Figure 8 is a graphical explanation of these parameters.

Exit temperature pattern factor  $\vec{\delta}$  is one parameter which is defined as

$$\overline{\delta} = \frac{T_{t4}, \max - T_{t4}}{T_{t4} - T_{t3}}$$

where  $T_{t4}$  and  $T_{t3}$  are averages of temperatures measured at the exit and inlet and where  $T_{t4, \max} - T_{t4}$  is the maximum temperature occurring anywhere in the combustor exit plane minus the average exit temperature. (Symbols are defined in appendix C.) This is a useful parameter for preliminary screening, but it does not take into account the desired radial temperature profile for which the combustor was designed. The desired average radial distribution of temperature at the combustor exit plane is determined by the stress and cooling characteristics of the turbine. For purposes of evalu-

ating the double-annular combustor, an exit radial temperature profile was selected for conditions that are typical of advanced engines.

The two other parameters take the design profile into account. These parameters are

$$\delta_{\text{stat}} = \frac{\left[ T_{\text{t4j(loc)}} - T_{\text{t4j(des)}} \right]_{\text{max}}}{T_{\text{t4}} - T_{\text{t3}}}$$
$$\delta_{\text{rot}} = \frac{\left[ T_{\text{t4j}} - T_{\text{t4j(des)}} \right]_{\text{max}}}{T_{\text{t4}} - T_{\text{t3}}}$$

where  $\begin{bmatrix} T_{t4j(loc)} - T_{t4j(des)} \end{bmatrix}_{max}$  for  $\delta_{stat}$  is the maximum positive temperature difference between the highest local temperature at any given radius and the design temperature for that same radius (subscript j refers to any radial location in the radial temperature profile) and where  $\begin{bmatrix} T_{t4j} - T_{t4j(des)} \end{bmatrix}_{max}$  for  $\delta_{rot}$  is the maximum temperature difference between the average temperature at any given radius around the circumference and the design temperature for that same radius (see fig. 8). The term  $T_{t4} - T_{t3}$  used in all three parameters is the average temperature rise across the combustor  $\Delta T$ .

The parameter  $\delta_{stat}$  is a measure of the quality of the exit temperature profile on the turbine stator, and  $\delta_{rot}$  is a measure of the quality of the exit temperature profile on the turbine rotor.

#### RESULTS AND DISCUSSION

The double-annular ram-induction combustor was tested at inlet-air pressures up to  $90 \text{ psia} (62.0 \text{ N/cm}^2)$ , inlet-air temperature up to  $1150^{\circ}$  F (894 K), and exit temperatures up to  $2200^{\circ}$  F (1476 K). The test data and performance of the combustor at design conditions are given in table III and for all other conditions in table IV. Combustion efficiency was 100 percent for the takeoff and cruise conditions. Exit temperature profiles were satisfactory and equalled or exceeded those for a single-annular ram-induction combustor that was 50 percent longer (ref. 4). Exit temperature pattern factors measured 0.20 and 0.25 for the cruise and takeoff conditions. These compare to 0.21 and 0.29 for the same conditions measured in the 50-percent longer single-annular combustor. Exit temperature radial profiles came very close to the design profile. The average circumferential

temperature at any of the five radii traversed was less than  $75^{\circ}$  F (42 K) above the design objective. The pressure loss was higher than desired, being 6.2 and 8.4 percent of the diffuser inlet total pressure for the takeoff and cruise conditions, respectively. For the same conditions in the single-annular combustor which had a larger open hole area (289 sq in. (1865 cm<sup>2</sup>)), the losses were 4.5 and 5.4 percent.

#### Total Pressure Loss

The combustor total pressure loss (overall diffuser-combustor) expressed as the percent of the average diffuser inlet total pressure is shown as a function of the diffuser inlet Mach number in figure 9. The two curves shown are data obtained without combustion (isothermally) and with a combustor temperature rise of approximately  $1100^{\circ}$  F (639 K). The test conditions were total pressures of 60 and 90 psia (41. 4 and 62. 0 N/cm<sup>2</sup>) and inlet air temperatures of  $600^{\circ}$ ,  $1050^{\circ}$ , and  $1150^{\circ}$  F (589, 839, and 894 K). The diffuser inlet velocity profile was uniform (±1 percent of average velocity) across the inlet annulus for these tests. The diffuser inlet Mach numbers for the sea level takeoff and cruise conditions are noted on the abscissa. At these conditions the total pressure loss of the double-annular combustor is considered to be high, compared to that of the state-of-the-art for longer combustors. The total pressure losses for the single-annular ram-induction combustor were about 1. 5 to 3. 0 percentage points below the values for the double-annular combustor at the takeoff and cruise conditions.

#### Exit Temperature Profile

The measured average radial temperature profile is compared with the design radial profile in figures 10(a) to (c) for the sea level takeoff, Mach 2.7 cruise, and Mach 3.0 cruise conditions. Also shown are the maximum temperatures measured at any point around the circumference for five radial positions. This provides an indication of the hot spots in the exit temperature. The agreement with the design profile is very good for all three conditions. The measured average temperature profile did not exceed the design profile by more than  $60^{\circ}$  F (33 K) at the maximum temperature rise (takeoff) condition. At the cruise conditions, the deviation was considerably less, not exceeding the design profile by more than  $40^{\circ}$  F (22 K) for the Mach 2.7 condition and  $30^{\circ}$  F (17 K) for the Mach 3.0 condition.

The double-annulus combustor and associated dual fuel systems allow adjustments to be made in the exit temperature profile. An example of this capability is shown in figure 11. An even fuel flow split in each annulus is compared with one in which the fuel split is greater in the inner annulus than in the outer annulus. An increase in the inner

to outer annulus fuel split ratio from 1.00 to 1.19 increased the exit temperatures toward the inner diameter and decreased the temperatures toward the outer diameter.

#### **Off-Design Combustion Efficiency**

The effect on efficiency of inlet-air pressures below 60 psia (41.4  $N/cm^2$ ) and a constant inlet-air temperature of 600<sup>0</sup> F (589 K) with variable fuel-to-air ratios and reference velocities is shown in figure 12. The following results were obtained:

(1) Efficiency dropped from 100 percent with low fuel-to-air ratios (0.010) at inletair pressure below 30 psia (20.6  $N/cm^2$ ) but held at 100 percent at higher fuel-to-air ratios (>0.015) at inlet air pressures down to 15 psia (10.3  $N/cm^2$ ) before dropping off.

(2) Efficiency improved substantially with increasing fuel-to-air ratio from 0.010 to 0.022 at pressures below 30 psia (20.6  $N/cm^2$ ).

(3) There was only a slight improvement of efficiency with reduced reference velocity (150 to 100 ft/sec (45.7 to 30.5 m/sec)) at various fuel-to-air ratios and pressures.

(4) The best efficiency at low pressure was 97 percent at a total pressure of 10 psia  $(6.9 \text{ N/cm}^2)$ , the reference velocity of 100 feet per second (30.5 m/sec), and a fuel-to-air ratio of 0.022.

The effect on efficiency of low inlet-air temperatures (below  $600^{\circ}$  F (589 K)) at a constant inlet-air pressure of 30 psia (20.6 N/cm<sup>2</sup>) with variable fuel-to-air ratios and reference velocities is shown in figure 13. The following results were obtained:

(1) Efficiency fell off rapidly at temperatures below  $600^{\circ}$  F (589 K) except for low reference velocities (100 ft/sec or 30.5 m/sec) and for fuel-to-air ratios of about 0.015 where efficiency fell off rapidly at temperatures below  $400^{\circ}$  F (477 K).

(2) At inlet-air temperatures below  $400^{\circ}$  F (477 K), operation at 100 percent efficiency with high fuel-to-air ratios (>0.015) was limited because of the onset of acoustic instability.

(3) The efficiency at low temperatures was 30 percentage points lower than that for the single-annular ram-induction combustor (53 percent compared to 83 percent at an inlet-air temperature of  $300^{\circ}$  F (422 K), an inlet-air pressure of 30 psia (20.6 N/cm<sup>2</sup>), a fuel-to-air ratio of 0.015, and a reference velocity of 150 ft/sec (45.7 m/sec)). The short combustor length which reduces residence time is probably the reason for the relatively lower efficiency at conditions of temperature and pressure which characteristically produce low efficiency. No attempt was made to improve the efficiency.

The data of figures 12 and 13 are used in the combustion efficiency correlating parameter  $P_{t3}T_{t3}/V_{ref}$  shown plotted in figure 14. This parameter was developed for combustor inlet variables with the simplifying assumptions noted in reference 5. Reasonable correlation is apparent between this parameter and efficiency for two constant

fuel-to-air ratios. Combustion efficiency drops from 100 percent at  $P_{t3}T_{t3}/V_{ref} = 30 \times 10^3$  pound-second-<sup>O</sup>R per cubic foot (2.62×10<sup>6</sup> N-K-sec/m<sup>3</sup>).

#### Combustor Exit Temperature Response to Rapid

Increase in Fuel Flow

Special tests were conducted to determine how well the combustor would react to rapid increase in fuel flow. For these transient tests, a special 0.005-inch (0.127-mm) diameter platinum - 13-percent-rhodium/platinum wire thermocouple was mounted in the combustor exhaust plane. The measured temperatures were corrected for a time constant of 0.0412 second determined for this thermocouple. The test conditions were for a 30-psia (20.6-N/cm<sup>2</sup>) combustor pressure, a 66-foot-per-second (20.1-m/sec) reference velocity, and a  $350^{\circ}$  F (451 K) inlet-air temperature.

The test was started with the combustor operating at an initial exit temperature of  $1500^{\circ}$  F (1090 K) as measured by the thermocouple. Several step increases in fuel flow were then made to produce temperatures up to about 2400° F (1590 K). At each step increase, a steady-state temperature and a corresponding fuel flow were measured. The transient response to a rapid increase in fuel flow was made through this fuel flow range by starting again at 1500<sup>°</sup> F (1090 K) initial temperature level and increasing the fuel flow in about 1.4 seconds to a value which should produce  $2400^{\circ}$  F (1590 K). The fuel flow increase during this period was converted to an exit temperature based on the steady-state fuel flow exit temperature relation determined from the initial steady-state step increases. This temperature increase is plotted against fuel burst time in figure 15 as a dashed line. The measured temperature increase corresponding to this same time interval is also plotted on this figure. The temperature increase, as shown in figure 15, lags behind that expected from the fuel flow increase. After about 3.4 seconds the temperature rise had caught up with the rapid increase in fuel flow. This lag in temperature may be caused by the low efficiency of the combustor at the low inlet-air pressure and temperature and the initial fuel-to-air ratio at which these tests were conducted (as shown in fig. 13). The combustor temperature response is considered within the controlled acceleration limits for an engine.

#### Combustor Blowout and Altitude Relight

To obtain blowout data, the combustor was first ignited at moderate inlet-air pressure and temperature. Then the pressure was lowered in steps while the inlet-air temperature, fuel-to-air ratio, and reference Mach number were held constant. After each

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change in pressure a fuel burst test (rapid increase in fuel-to-air ratio) was made to determine whether the combustor would produce a corresponding temperature rise. This procedure indicated whether an engine would accelerate at these conditions. The fuelto-air ratio would then be returned to the original value, the pressure reduced a further step, and the procedure repeated. Finally, at some pressure level, the combustor would blow out. Relight was then attempted at successively increased pressure levels. If relight occurred, a fuel burst test was conducted at that condition to determine again if engine acceleration would occur. These procedures were repeated at a lower inletair temperature.

The facility limited testing to a minimum inlet-air temperature of  $65^{\circ}$  F (292 K). Tests were conducted at reference Mach numbers of 0. 1, 0.075, and 0.050 with a fuelto-air ratio of 0.01. This corresponded to a temperature rise of about  $700^{\circ}$  F (389 K). The fuel burst tests increased the fuel-to-air ratio sufficiently to give a temperature rise of  $1000^{\circ}$  F (555 K).

Figure 16 presents the results of these tests. The limit lines shown are for combustor blowout noted by the solid symbols. Relight followed by successful temperature rise is shown by the open symbols. A significant effect of reference Mach number is apparent. The lower reference Mach numbers (<0.1) significantly improved the blowout and, to some extent, the relight characteristics of the combustor. At an inlet-air temperature of  $75^{\circ}$  F (297 K), blowout occurred at inlet-air pressures as low as 9 psia (6.2 N/cm<sup>2</sup>). This was obtained, however, only at the low reference Mach number of 0.05. Under these same conditions, relight was possible only at inlet-air pressures of 14 psia (9.6 N/cm<sup>2</sup>). The blowout limit line at a reference Mach number of 0.05 follows about the same trend with pressure and temperature as that for relight in the single annular combustor at a higher Mach number of 0.10. No attempt was made to improve the relight characteristics of the double-annular combustor.

#### Durability

A total of about 100 hours of operation was accumulated on the combustor during the performance evaluation testing. The combustor showed deterioration after approximately 1 hour of operation at the most severe Mach 3.0 cruise condition of an  $1150^{\circ}$  F (894 K) inlet-air temperature and a 90-psia (62-N/cm<sup>2</sup>) inlet-air pressure. Some burning of the center scoops and inner lip of the outside headplate occurred. At the Mach 2.7 cruise condition ( $1050^{\circ}$  F (839 K) inlet-air temperature and 60-psia ( $41.2-N/cm^{2}$ ) inlet-air pressure), however, very little deterioration occurred after 15 hours of operation. The center scoops did not burn, and the headplate areas were affected only slightly. No durability problems were evident for the sea-level takeoff condition.

#### Smoke Density Evaluations

Measurements were made to determine the general level of smoke density in the combustor exhaust gases at design operating conditions.

Smoke numbers (defined in appendix B) obtained at conditions simulating takeoff, Mach 2.7, and 3.0 cruise were 4 or less. These values are very low, being well below the threshold of visible smoke. The very low smoke number at the simulated takeoff pressure of 90 psia ( $62 \text{ N/cm}^2$ ) would probably prevent reaching the visible smoke level at the higher normal takeoff pressure of 180 psia ( $124 \text{ N/cm}^2$ ).

#### SUMMARY OF RESULTS

A full-scale short-length turbojet combustor designed for Mach 3 cruise conditions was tested at the Lewis Research Center. This high performance short length combustor was achieved by using both the double annulus and the ram-induction concepts. The following results were obtained:

1. Combustor performance except for durability at the Mach 3.0 cruise conditions was satisfactory. The performance equalled or exceeded that of a similar ram-induction combustor designed for the same supersonic conditions having a single annulus and being 50 percent longer.

2. The combustor total pressure loss, including the diffuser, was 6.1, 8.4, and 9.1 percent at the simulated sea-level takeoff, Mach 2.7 cruise, and Mach 3.0 cruise conditions, respectively. At these conditions, the diffuser inlet Mach numbers were 0.249, 0.300, and 0.313, respectively. These pressure losses are about 1.5 to 3.0 percentage points greater than those of the single annular combustor for the same range of conditions.

3. Combustion efficiencies were 100 percent for the simulated takeoff and cruise conditions. At off-design conditions (low inlet-air temperature and pressures) the combustion efficiency was considerably less (53 percent at  $300^{\circ}$  F (923 K), 30 psia (20.6 N/cm<sup>2</sup>), 0.015 fuel-to-air ratio, and reference velocity of 150 ft/sec (45.7 m/sec)).

4. The exit radial average temperature profile was in good agreement with the design profile. The largest positive temperature differences between the circumferentially average temperature on any radius and the design temperature for that same radius were only  $60^{\circ}$  F (33 K) and  $30^{\circ}$  F (17 K) for the sea-level takeoff and Mach 3 cruise conditions, respectively.

5. Exit temperature pattern factors were 0.20 to 0.25 at all design conditions. These compare to 0.21 and 0.29 for the same conditions in a single annular combustor 50 percent longer. 6. Combustor exit temperature response to a rapid step increase in fuel-to-air ratio was good. The temperature increased not quite as rapidly as the fuel flow during the 3.4 seconds when the temperature rise was increased from  $1500^{\circ}$  F (1090 K) to  $2400^{\circ}$  F (1590 K). The combustor response is considered to be within the controlled acceleration limits for an engine.

7. Combustor blowout limit was 9.0 psia (6.2 N/cm<sup>2</sup>) at 75<sup>o</sup> F (297 K) at a reference Mach number of 0.05 and a fuel-to-air ratio of 0.01. The minimum pressure for blowout increased sharply at temperatures near 75<sup>o</sup> F (297 K). The minimum pressure for relight at these same conditions was 14 psia (9.6 N/cm<sup>2</sup>). No attempt was made to improve the relight characteristics of this combustor.

8. Durability evaluated during 100 hours at variable operating conditions (including 15 hr at Mach 2.7 cruise condition) was satisfactory except at the Mach 3.0 cruise condition. Here burning of center scoops and liner areas near the headplate occurred in less than 1 hour of operation.

9. Exhaust gas smoke numbers obtained to evaluate smoke density at the operating conditions measured 4 or less. These values are well below the threshold of visible smoke.

Lewis Research Center,

National Aeronautics and Space Administration,

Cleveland, Ohio, December 1, 1970,

720-03.

#### APPENDIX A

#### TEST FACILITY

The full-scale double-annular ram-induction combustor investigation was conducted in a closed-duct test facility of the Engine Components Research Laboratory at Lewis. A sketch of this facility is shown in figure 17. Airflows for combustion up to 300 pounds per second (136 kg/sec) at pressures from below atmospheric to 10 atmospheres could be heated to  $1200^{\circ}$  F (922 K) without vitiation before entering the combustor under test.

Figure 18 shows the combustor test section and the connected inlet and outlet ducting. About  $4\frac{1}{2}$  pipe diameters of constant-area duct was ahead of the test section. Following the inlet ducting was the combustor housing which included the diffuser inlet and diffuser as part of the housing. The combustor housing measured 42 inches (1.07 m) at the maximum diameter and was 37.75 inches (0.96 m) long including the inlet section. Following the combustor housing was the outlet instrumentation section. At this section and downstream, the combustor exhaust gases were cooled by a water-injection spray system. The exposed surfaces downstream of the combustor were cooled by two methods: (1) circulating water in passages adjacent to the hot surfaces and (2) water sprays impinging directly on the exposed surfaces. Photographs of the test section installation are shown in figure 19.

For combustor inlet-air temperatures of  $600^{\circ}$  F (589 K) or less, an indirect-fired heat exchanger was used. For higher inlet temperatures (to  $1200^{\circ}$  F, 922 K), a second-stage of indirect heating was used. Heat for the second stage was provided by a natural-gas-fueled J-57 jet engine with an afterburner. The engine exhaust gases were passed through a heat exchanger of special design shown in figure 20.

Airflow rates and combustor pressures were regulated by remotely controlled valves upstream and downstream of the test section. Flow straighteners were used to evenly distribute the airflow entering the combustor (see fig. 17).

#### APPENDIX B

#### **INSTRUMENTATION**

Measurements to determine combustor operation and performance were recorded by the Lewis Central Automatic Data Processing System (ref. 6). Control room readout instrumentation (indicating and recording) was used to set and monitor the test conditions and the operation of the combustor. Pressures were measured and recorded by the central digital automatic multiple pressure recorder (DAMPR) and by strain-gage pressure transducers (ref. 7). Temperatures were measured by iron-constantan and Chromel-Alumel thermocouples for low- and medium-temperature conditions and by platinum -13-percent-rhodium/platinum thermocouples for high temperatures. The indicated readings of all thermocouples were taken as true values of the total temperatures. The platinum - 13-percent-rhodium/platinum thermocouples were of the high-recovery aspirating type (ref. 8, type 6).

Airflow rates were measured by square-edged orifices installed according to ASME specifications. Fuel flow rates were measured by turbine flowmeters using frequency-to-voltage converters for readout and recording.

The locations of the combustor instrumentation stations axially along the test section are shown in figure 18. Instrumentation at inlet station 3 is shown in figure 21. Inlet-air temperature was measured by eight Chromel-Alumel thermocouples that were equally spaced around the inlet at station 3. Inlet air total pressure was measured by eight five-point total pressure rakes equally spaced around the inlet at station 3. The pressure rakes measured the total pressure profile at centers of equal areas across the inlet annulus. Static pressure at the inlet was measured by 16 wall static pressure taps with eight on the outside and eight on the inside walls of the annulus.

Combustor outlet total temperature and pressure at instrumentation station 4 were measured at  $3^{\circ}$  increments around the exit circumference. At each  $3^{\circ}$  increment, five temperature and pressure points were measured across the annulus. These points were located at centers of equal areas across the annulus. The water-cooled probe assembly containing the five temperature and pressure sensors is shown in figure 22. Three of these probes, each on an arm  $120^{\circ}$  apart, rotated  $120^{\circ}$  providing full coverage of the circumference. Water-cooled shields protected these probes when they were not in use at three fixed points in the exhaust stream. At these points, temperature and pressure were not measured. The probes were made of platinum-rhodium alloy where exposed to the hot exhaust gases. Also located at station 4 were eight wall static pressure taps.

A schematic of the equipment used to obtain the smoke density or smoke number (SN) is shown in figure 23. Combustor exhaust gas samples were picked up by the aspirating thermocouple probes and cooled to room temperature by a heat exchanger. A portion of these gases, which was used to obtain the smoke number, was piped into a plenum chamber. The chamber was continuously purged by these gases and maintained at about 2 psi  $(1.4 \text{ N/cm}^2)$  above atmospheric pressure. A Von Brand Smokemeter was used to obtain the smoke stain on Whatman No. 4 filter tape. To obtain a smoke stain, the vacuum pump was started, and the gas flow adjusted to 0.6 standard cubic feet per minute  $(2.83 \times 10^{-4} \text{ m}^3/\text{sec})$  as measured by the calibrated rotameter. A pressure drop across the filter tape of about 5 inches (12.7 cm) of mercury was maintained for all the tests. The tape speed was 4 inches per minute (0.17 cm/sec) which gave a sample flow rate of 0.3 standard cubic feet per minute per square inch  $(2.19 \times 10^{-5} \text{ m}^3/\text{sec}-\text{cm}^2)$  of tape. Two moisture traps were used between the plenum chamber and the heated head on the smokemeter. During a test, some moisture was carried into the first trap; however, none was ever evident in the second trap.

The absolute reflectivity of the stained filter tape was measured with a Welsh Densichron using a gray background. The Densichron was calibrated with a Welsh Gray Scale.

The smoke number was determined from the following equation:

$$SN = 100 \left| 1 - \left( \begin{array}{c} Percent absolute reflectivity of sample \\ Percent absolute reflectivity of clean paper \end{array} \right) \right|$$

### APPENDIX C

### SYMBOLS

C <sub>d</sub>	discharge coefficient	${}^{\Delta T}$ rot	$\begin{bmatrix} T_{t4j} - T_{t4j(des)} \end{bmatrix}$
F/A	fuel-to-air ratio	$\Delta \mathbf{T}$	temperature rise across
P <sub>t</sub>	total pressure		combustor ( $T_{t4} - T_{t3}$ )
т <sub>t</sub>	total temperature	$^{\mathrm{P}}\mathbf{s}$	static pressure
w <sub>a</sub>	mass airflow rate	SN	smoke number
u V <sub>ref</sub>	reference velocity	Subscrip	ts:
<sup>∙</sup> ref ∆P	total pressure drop across	ref	reference
	combustor $(P_{t3} - P_{t4})$	3	diffuser inlet
М	Mach number	4	combustor exit
W <sub>f</sub>	mass fuel flow rate	j	radial location
$^{\Delta T}$ stat	$\begin{bmatrix} T_{t4j(loc)} - T_{t4j(des)} \end{bmatrix}$		

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#### TABLE I. - DOUBLE-ANNULAR RAM-INDUCTION

#### COMBUSTOR DIMENSIONS AND SPECIFICATIONS

Lengths	1
Compressor exit to turbine inlet, in. (cm)	20. 25 (51. 5)
Fuel nozzle face to turbine inlet, in. (cm)	12.00 (30.5)
Diameters	
Inlet outside diameter,in.(cm)	31.80 (80.77)
Inlet inside diameter, in. (cm)	28.00 (71.1)
Outlet outside diameter, in. (cm)	35.38 (89.9)
Outlet inside diameter, in. (cm)	27.50 (69.9)
Shroud	
Outside diameter, in. (cm)	37.09 (94.2)
Inside diameter, in. (cm)	22.52 (57.2)
Reference area (between shrouds), in. $^2$ (cm $^2$ )	663 (4270)
Diffuser inlet area, in. $^2$ (cm $^2$ )	182.5 (1177)
Open hole area (including cooling), in. $^2$ (cm $^2$ )	200 (1291)
Flow spreader inlet areas, in. $2 (\text{cm}^2)$	
Outside diameter passage, in. $^2$ (cm <sup>2</sup> )	54.0 (348)
Center passage, in. <sup>2</sup> (cm <sup>2</sup> )	121.5 (785)
Inside diameter passage, in. $^2$ (cm $^2$ )	52.5 (339)
Exit area, in. $^2$ (cm $^2$ )	388 (2503)
Number of fuel nozzles and swirlers	64
Number of diffuser struts	16
Number of ram-induction scoops	512
Rows, primary zone	1
Rows, secondary zone	1
Ratio length to annulus height	
Outer annulus	4.8
Inner annulus	3, 9

### TABLE II. - SCOOP AREAS<sup>a</sup> AND SIZES FOR DOUBLE-ANNULAR

Type of scoop	Dischar	ge area	Ler	igth	Width		
	in. <sup>2</sup>	cm <sup>2</sup>	in.	cm	in.	cm	
Outer liner primary	12.480	80.516	0.458	1. 163	0.458	1. 163	
Outer liner secondary	22. 144	142.864	. 614	1.560	. 614	1. 560	
Outer centershroud primary	12.480	80.516	. 458	1.163	. 458	1. 163	
Outer centershroud secondary	11.072	71.432	.614	1.560	. 306	. 803	
Inner centershroud primary	12.480	80.516	.458	1.163	.458	1.163	
Inner centershroud secondary	11.072	71.432	. 614	1.560	. 306	. 803	
Inner liner primary	12.544	80.929	.481	1.222	.438	1.112	
Inner liner secondary	22.720	146.580	. 773	1.963	.490	1. 245	

### RAM-INDUCTION COMBUSTOR<sup>b</sup>

<sup>a</sup>All areas are actual area for a full annulus. <sup>b</sup>See fig. 3.

#### TABLE III. - COMBUSTOR PERFORMANCE DATA FOR DOUBLE-ANNULAR RAM-INDUCTION

#### COMBUSTOR-PERFORMANCE AT DESIGN CONDITIONS

	Inlet air conditions												Combustor operating conditions				
Engine condition	Total pressure, P <sub>t3</sub>							Air flow, W <sub>a</sub>				Reference velocity, V <sub>ref</sub>		Fuel- air	Average exit temperature		
	psia	N/c	m <sup>2</sup>	٥ <sub>F</sub>	к	:	lbm/se	n/sec kg/sec		Mach number, M <sub>3</sub>		ft/sec m/sec		ratio, <sup>W</sup> f	°F	t4 K	
Takeoff Mach 2.7 cruise Mach 3.0 cruise	89.7 61.8 89.2	42.	.6	602 1053 1147	59 84 89	1	111.2 73.4 104.4	3	0.4 3.3 7.4	0, 252 , 296 , 302	3	105.5 144.2 150.8	32.2 44.0 46.0	0.0250 .0186 .0173	2219 2203 2192	1488 1479 1473	
Engine condition	ine condition Maximum Exit temperature p						ombustor performance characte parameters Tempera ture rise arameters across				era- ise	1	$-\frac{P_{t3}T_{t3}}{V_{ref}}$				
	$\begin{bmatrix} temp \\ atur \\ (T_{t4}) \end{bmatrix}$	е,	factor, δ	<sup>õ</sup> stat	∆T <sub>sta</sub> <sup>0</sup> R	t(max) K	<sup>ð</sup> rot	ΔT <sub>ro</sub>	t(max) K	comb tor ΔT <sup>O</sup> R		sure loss, ∆P/P <sub>t3</sub> , percent	ciency, percent	lbs- <sup>0</sup> R-s ft <sup>3</sup>		K-sec m <sup>3</sup>	
Takeoff Mach 2.7 cruise Mach 3.0 cruise	2630 2440 2447	1717 1611 1615	. 206	0. 249 . 202 . 2 <b>3</b> 9	403 232 250	224 129 139	0.044 .024 .027	72 25 28	40 14 15	1617 1150 1045	898 639 581	8. 24	104.2 102.1 100.6	130. 2≁1 93. 4 136. 9	8	35×10 <sup>6</sup> 14 92	

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#### TABLE IV. - COMBUSTOR PERFORMANCE DATA FOR

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	-т	-	Inlet air c	onautions			,		mbustor op	erating Co 1	unaition i	s
Run	Total pressure,		Total tem	perature,	Air	low,	Diffuser	Referenc	e velocity,	Fuel-	Avera	ge
no.	{ 1	P <sub>t3</sub>	T <sub>t</sub>	3	w	a	inlet	v,	ref	air	tempe	rat
			° <sub>F</sub>	Ì		Î I	Mach		·····	ratio,	1	r <sub>t4</sub>
	psia	N/cm <sup>2</sup>	F	к	lbm/sec	kg/sec	number, <sup>M</sup> 3	ft/sec	m/sec	w <sub>f</sub>	°F	
		-										
222	10.0	6.9	584	580	12, 8	5.8	0.258	108.0	32.9	0.0092	1055	
223	10.1	7.0	581	578	12.9	5.8	. 254	106.4	32.4	. 0149	1486	1
224	10.1	7.0	576	575	12.8	5.8	. 252	105.2	32.1	. 0203	1829	1
214	14.9	10.3	598	588	18.0	8.1	. 243	102.7	31.3	.0100	1210	
215	15.2	10.5	598	588	18.0	8.2	. 239	100.8	30.7	.0158	1631	1
216	15.2	10,5	594	585	17.9	8.1	. 238	100.3	30.6	. 0220	1983	1
208	20.0	13.8	598	588	23.5	10.7	. 237	100.2	30.5	.0100	1245	
209	20.0	13.8	598	588	23.5	10.7	. 236	99.9	30.4	.0163	1674	
210	19.9	13.7	596	586	23.5	10.6	. 237	100.0	30.5	. 0228	2047	
203	25.1	17.3	600	589	29.1	13.2	. 234	99.0	30.2	.0101	1280	!
204	25.1	17.3	607	592	29.2	13.2	. 235	99.5	30.3	.0164	1697	1
205	25.1	17.3	606	592	29,2	13.2	. 235	99.7	30.4	.0217	2005	
199	40.0	27.6	597	587	47.0	21.3	. 238	99.7	30.4	.0100	1289	
249 250	40.0 50.3	27.6 34.7	607 607	593 593	47.5 58.8	21.5 26.7	. 240 . 237	101, 5 100. 0	30.9 30.5	.0160 .0101	1680 1294	1:
						-						
252	50.0	34.4	599	588	58.9	26.7	. 238	100.1	30.5	.0162	1686	1
255	60.3	41.6	595	586	69.7 69.5	31.6	. 233	97.8	29.8	.0102	1299	9
256	60.5	41.7	602 504	590 585	69.5	31, 5 8, 2	. 232	98.0 152.3	29.9 46.4	.0164 .0099	1705 1089	12
217 218	9.9 10.1	6.9 7.0	594 591	585	18.0 18.0	8.2	. 385 . 377	132.3	40.4	. 0158	1505	10
001	10.0	6.9	596	586	17.8	8.1	. 381	151.4	46.1	. 0220	1859	12
221 211	15.0	0.9 10.4	599	588	26.4	12.0	. 373	131.4 149.0	45.4	.0101	1208	9
211	15.0	10. 4	599	588	26.5	12.0	. 369	147.6	45.0	.0163	1640	11
213	15.0	10.3	602	590	26.5	12.0	. 376	150.3	45.8	. 0223	1984	13
206	20.0	13.8	604	591	34,7	15.7	. 367	147.6	45.0	. 0101	1254	9
207	19.9	13.8	601	589	34.5	15.7	. 366	146.9	44.8	. 0166	1683	11
202	24.9	17.1	601	589	44.8	20.3	. 385	152.7	46.5	. 0099	1250	9
245	24.9	17.2	603	590	46.8	21.2	. 405	159.2	48.5	.0152	1596	11
227	30.0	20.7	597	587	54.0	24.5	. 383	151.7	46.2	. 0099	1258	9
247	39.8	27.5	608	593	70.6	32.0	. 379	151.1	46.1	. 0102	1293	9
253	50.2	34.6	580	578	88.6	40.3	. 373	147.0	44.8	. 0100	1254	9
229	30.0	20.7	87	303	65.6	29.8	. 324	95.6	29.1	. 0104	384	4
230	29.7	20.5	87	304	65.1	29.5	. 325	95.8	29.2	.0149	541	5
232 233	29.9	20.6	285 291	414 417	50.2 49.8	22.8 22.6	. 288 . 288	100.4 100.9	30.6 30.8	. 0100	837	7
	29.7	20.5		ļ	l			-		.0135	1136	
236	30.0	20.7	402	479	44.2	20.0	. 270	101.9	31.1	. 0099	1023	8
237 238	30.1	20.7	405 411	481 484	44.0 44.0	19.9 20.0	. 268 . 270	101.4 102.4	30.9 31.2	.0159 .0216	1479 1849	10 12
238 200	30.0 29.9	20.7 20.6	613	484 596	44.0 36.6	16.6	. 270	102.4	32.1	. 0097	1269	9
200	30.2	20.8	607	590 592	36, 5	16.6	. 247	103.9	31.7	. 0156		11
244	30.1	20.7	605	592	37.3	16,9	. 252	106.1	32.3	. 0209	1952	13
234	29.7	20. 7	292	417	74.2	33.7	. 466	149.2	45.5	.0102	713	6
235	30.0	20.7	296	420	74.4	33.7	. 463	149.1	45.4	.0158	807	7(
239	29.9	20.6	411	484	67.7	30.7	. 451	156.9	47.8	. 0098	977	79
240	30.1	20.7	406	481	67.2	30.5	. 443	154.0	46.9	.0129	1153	8
241	30.1	20, 8	612	595	55.7	25.3	. 400	158.1	48.2	. 0153	1621	

#### DOUBLE-ANNULAR RAM-INDUCTION COMBUSTOR

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#### Combustor performance characteristics

Maximum	n Exit temperature parameters							Temp	1	Combus-	Combus-	РТ	
exit		Exit temperature parameters					ture r		tor	tion	$P_{t3}T$	13	
hot spot	Pattern		Prof	ile pa	rameter	s		acro	ss	pres-	effi-	Vre	
temper-	factor,	<sup>δ</sup> stat	∆T <sub>stat</sub>		<sup>δ</sup> rot	ΔT <sub>rot</sub>		comb	us-	sure	ciency,	lbs- <sup>0</sup> R-sec	
ature,	δ	stat		(max)	rot	rot	(max)	tor	,	loss,	percent	ft <sup>3</sup>	m <sup>3</sup>
$\left( T_{t4} \right)_{max}$			°R	к		°R	к	$\Delta T$		$\Delta P/P_{t3}$ ,		i	
								ο <sub>R</sub>	к	percent			
<sup>o</sup> f K										ļ			
1224 935	0, 359	0.482	227	126	0.134	63	35	471	262	7.17	76.1	$13.9 \times 10^{3}$	1.21×10 <sup>6</sup>
1748 1226	. 290	. 237	214	119	. 042	38	21	904	502	7.11	92.9	14.2	1.25
2173 1463	. 274	. 226	284	158	.033	41	23	1253	696	7.27	97.2	14.4	1.25
1376 1020		. 275	168	94	. 054	33	18	612	340	6.09	91.0	22.1	1.93
1934 1330	. 294	. 245	254	141	.023	23	13	1033	574	6.01	100.7	22.9	2.00
2404 1591	. 303	. 260	361	201	. 056	78	43	1389	772	6.05	100.4	23.1	2.00
1407 1037	. 251	. 252	163	90	. 032	21	11	647	359	5.77	96.5	30.3	2,65
2003 1368	. 307	. 261	281	156	. 031	33	18	1076	598	5,85	102.0	30.5	2.66
2553 1674		. 315	458	254	.071	104	58	1451	806	5.97	101.7	30.3	2.64
1459 1066	. 265	. 236	160	89	. 022	15	8	679	377	5,45	100.6	38.6	3, 38
2010 1372	. 287	. 265	289	161	. 035	38	21	1090	606	5.68	102.9	38.7	3, 38
2459 1622	. 325	. 290	405	225	. 068	95	52	1399	777	5.74	102.8	38.6	3, 37
1460 1066		. 231	160	89	. 020	14	8	692	385	5.47	103.3	61.0	5, 33
2025 1380		. 317	340	189	. 025	27	15	1073	596	5.61	103.0	60.4	5.29
1499 1088	. 300	. 293	201	112	. 022	15	8	687	382	5.44	101.5	77.0	6.75
2023 1379	. 310	. 306	332	185	. 026	29	16	1087	604	5.66	103.4	76.0	6.64
1508 1093	. 298	. 291	205	114	. 021	15	8	704	391	5,28	102.6	93.7	8.17
2023 1379	1	. 283	312	174	.026	28	16	1103	613	5.30	103.7	94.5	8.23
1251 950		. 4 19	207	115	. 100	50	28	495	275	15.34	74.6	10.0	. 86
1798 1254	. 320	. 271	248	138	.050	46	25	914	508	14,96	89.0	10.2	. 89
2274 1519	. 328	. 289	365	203	.040	51	28	1263	702	15.63	91.3	9.9	. 87
1384 1024	J	. 295	180	100	. 053	32	18	609	339	14.00	90.5	23.9	1.34
1921 1323		. 261	272	151	. 024	25	14	1041	579	13.89	98.7	15.8	1.37
2372 1573	. 281	. 257	356	198	. 032	44	24	1382	768	14.59	99.1	15.3	1.33
1431 1050	. 282	. 253	164	91	. 034	22	12	650	361	13.23	96.2	20.8	1.81
1999 1366	. 292	. 277	300	167	. 029	32	18	1083	601	13.36	100. 9	20.6	1,81
1418 1043	1	. 252	163	91	. 032	21	11	649	361	14.40	97.9	24.8	2.17
1596 1142		. 272	270	150	. 028	27	15	993	552	16.07	100.1	24.0	2.09
1421 1045		. 240	159	88	.026	17 14	9	661	367 381	14.00 13.58	59.8 59.9	30.1	2.63
1493 1085	. 291	. 204 1	195	108	.041	14		685	001	10.00	00.0		
1459 1066		. 297	200	111	. 019	13	7	674	374	13.01	99.4	51.2	4.46
523 546	1	. 588	175	97	. 150	45	24	297	165	10.01	39.3	24.8	2.16
816 709		. 498	226	126	. 031	14	8 26	454	252 307	10.33	43.2	24.4 32.0	2.13
1085 836		. 403	223 264	124 146	.083	46	31	552 845	470	8.19 8.53	90.8	31.9	2.79
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1231 940		. 360	223	124	. 057	35	20	621	345	7.19	90.2	36.6	3. 19
1825 1270		. 267	287	159	. 032	34	19	1073	596		100.6	36.9	3.22
2389 1583	1	. 334	480 150	267 83	. 049	71 13	40 7	1439 656	799 365	7.57 6.09	102.8 101.3	36.7 43.8	3.20 3.83
1955 1341		. 229	278	154	.020	33	18	1044	580	6.09	101.3	43.8	3.89
-	ł	ļ			1				ļ	1			
2390 1583		. 281	378	210	. 045	60	33	1347	748		101.6	43.5	3.80
875 741	1	. 536	226	126	. 060	25	14	421	234	20.34	58.5	21.5	1.88
1042 834 1135 886		. 363	186 215	103 120	. 006	3 28	2	511 566		20.51 19.15	47.6 83.4	21.9 23.9	1.91 2.09
1388 1027		. 271	202	1120	.049	23	13	747	415	1	85.0	24.3	2.03
1919 1322		. 269	271	151	. 024	25	14	1009	561		101.1	29.4	2.57
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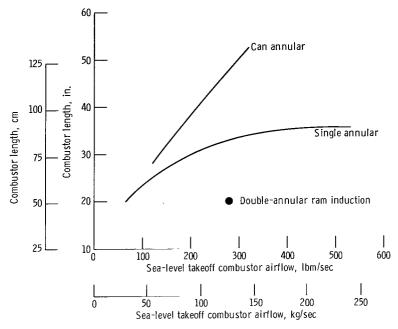


Figure 1. - Comparison of short-length ram-induction combustor with conventional combustors.

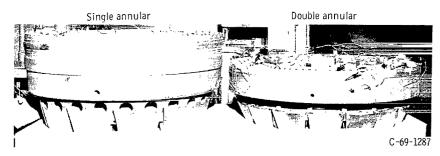


Figure 2. - Short-length double-annular combustor compared to length of single annular combustor for same operating conditions. (Exit transition liners removed).

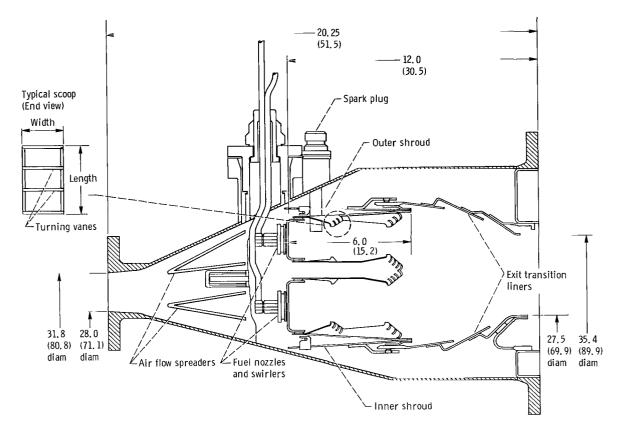
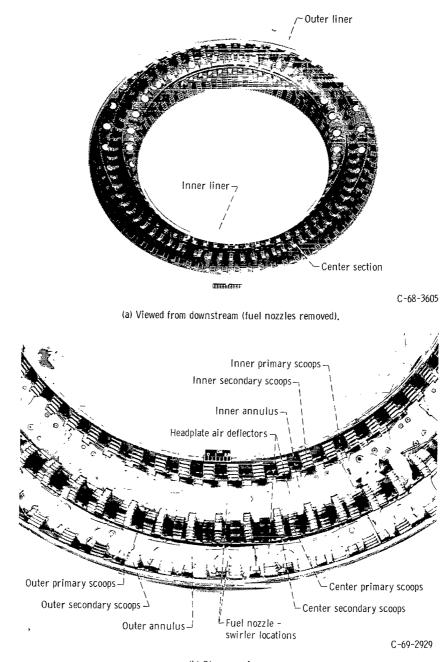


Figure 3. - Cross-section of double-annular ram induction combustor. Dimensions are in inches (cm).



(b) Close-up view. Figure 4. - Double-annular ram-induction combustor.

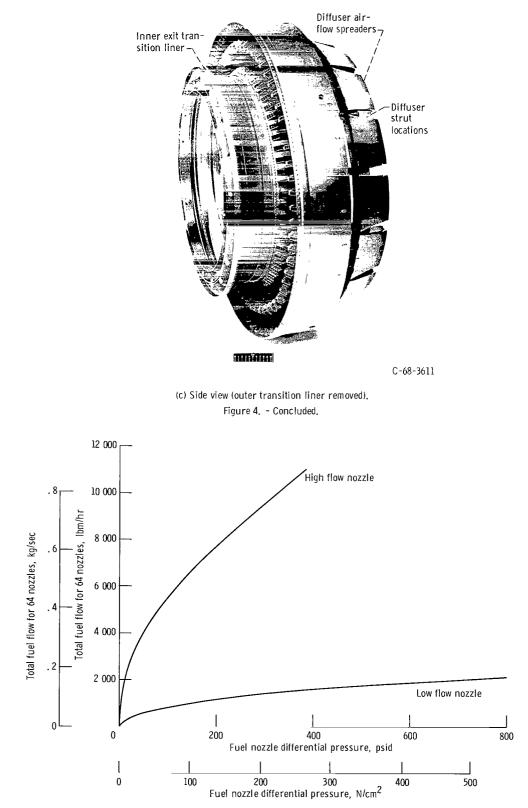


Figure 5. - Fuel flow as function of pressure drop across fuel nozzles of double-annular combustor.

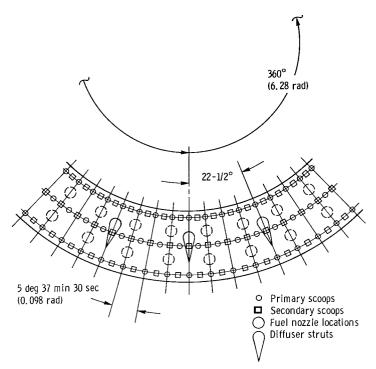


Figure 6. - Circumferential arrangement of combustor scoops and fuel nozzles.

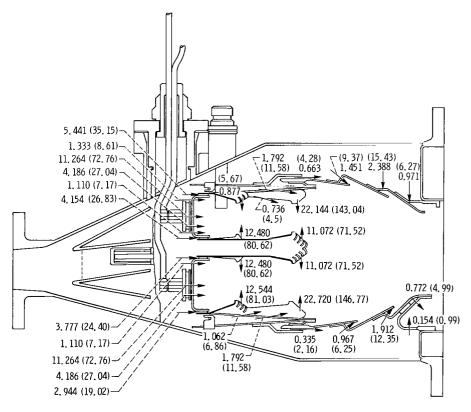
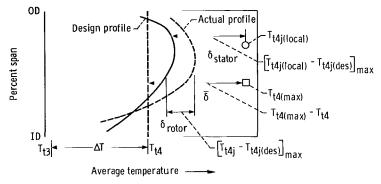
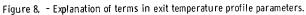


Figure 7. - Effective flow area distribution for double-annular ram-induction combustor. Swirler discharge coefficient, 0.50; hole discharge coefficient, 0.62; scoops and slot discharge coefficient, 1.00; total area, (effective), 183.374 square inches (1184.547 cm<sup>2</sup>). All areas are based on a full annulus with units of square inches (cm<sup>2</sup>).



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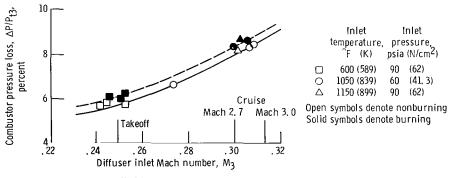
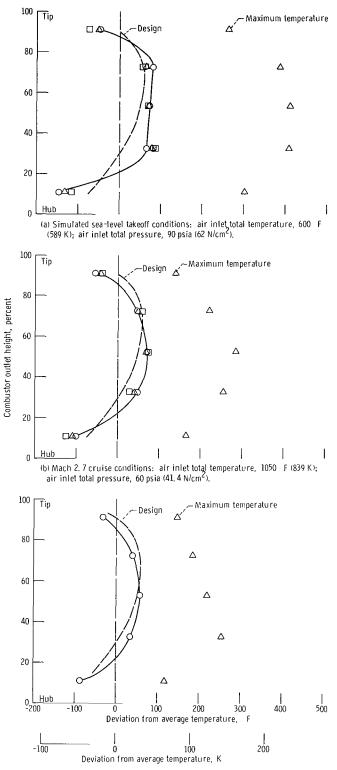


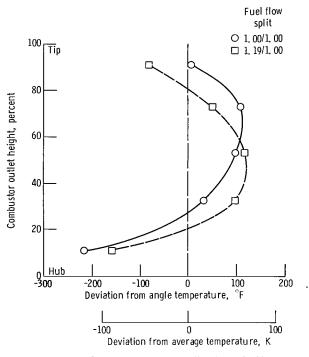
Figure 9. - Overall diffuser-combustor pressure loss as function of diffuser inlet Mach number.

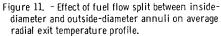


(c) Mach 3. O cruise conditions: air inlet total ten perature,  $1150^{\circ}$  F (894 K); air inlet total pressure, 90 psia (62 N/cm²).

Figure 10. - Average radial exit temperature profiles. Exit average temperature, 2200  $^\circ$  F (1478 K).

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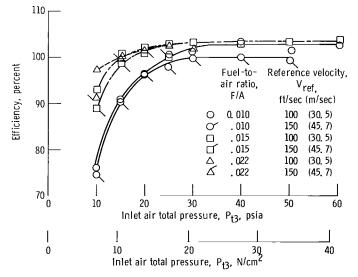


Figure 12. - Combustion efficiency at low inlet-air pressures for variable fuel-to-air ratios and reference velocities. Inlet-air temperature,  $600^{\circ}$  F (589 K).

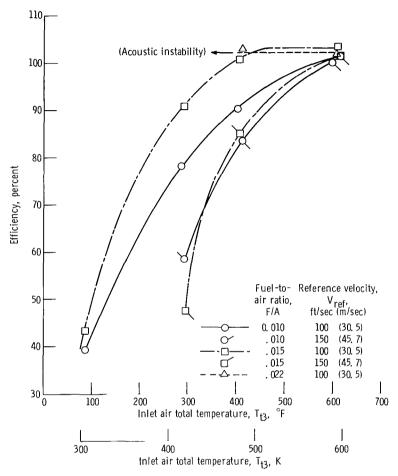
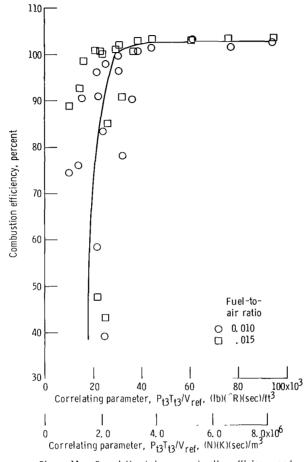
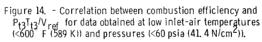


Figure 13. - Combustor efficiency at low inlet-air temperatures for variable fuelto-air ratios and reference velocities. Inlet-air pressure, 30 psia (20.6 N/cm<sup>2</sup>).





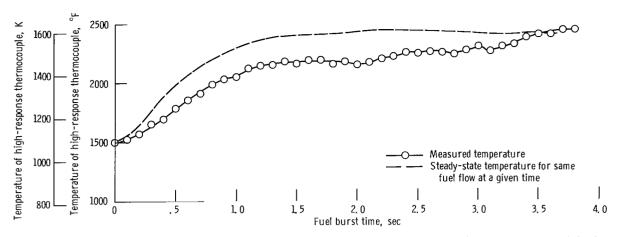


Figure 15. - Temperature response of double-annular combustor to rapid fuel flow increase. Exit temperature measured at only one spot in exit plane. Inlet-air temperature, 350° F; inlet-air total pressure, 30 psia (20.6 N/cm<sup>2</sup>); reference velocity, 66 ft/sec (20 m/sec).

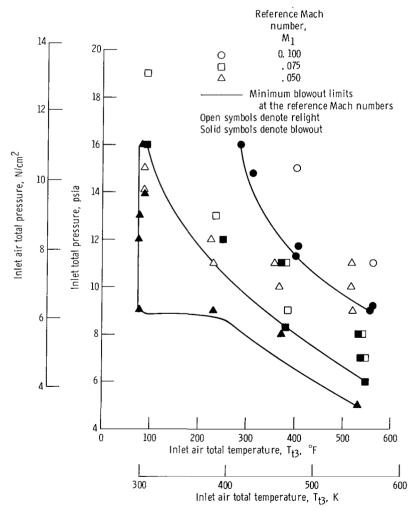


Figure 16. - Blowout and altitude relight characteristics of double-annular raminduction combustor.

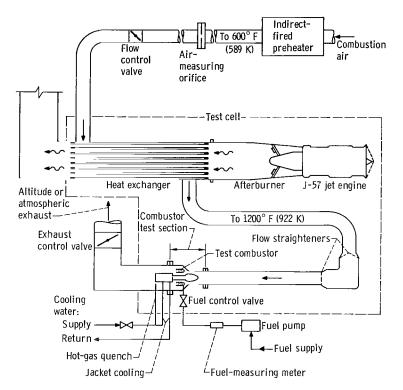


Figure 17. - Test facility and associated systems for full-scale advanced annular combustor tests.

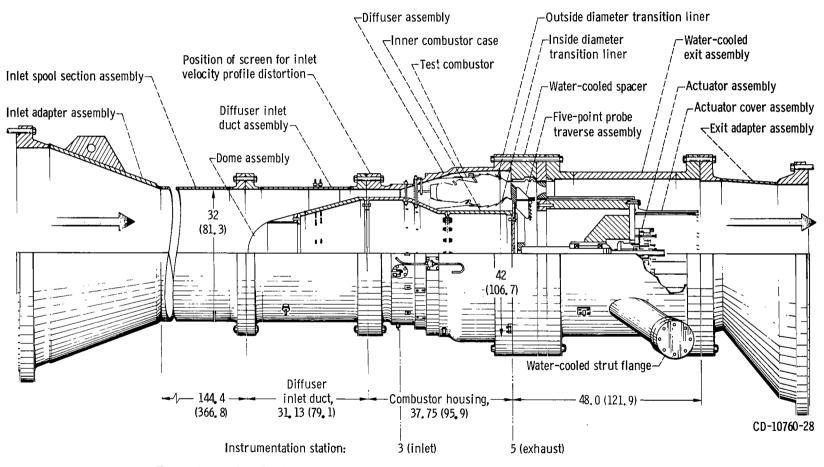
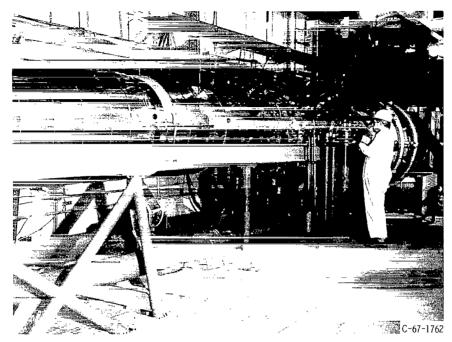
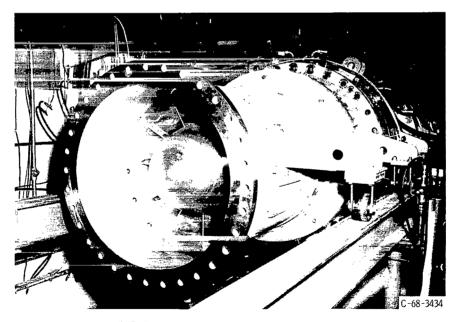


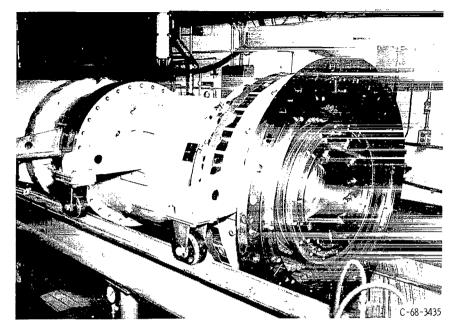
Figure 18. - Test section for full-scale advanced annular combustor. (Dimensions are in inches (cm).)



(a) Overall view looking downstream.



(b) Test section inlet (upstream pipe section removed). Figure 19. - Test section for advanced annular combustor.



(c) Test section outlet (downstream pipe section removed). Figure 19. - Concluded.

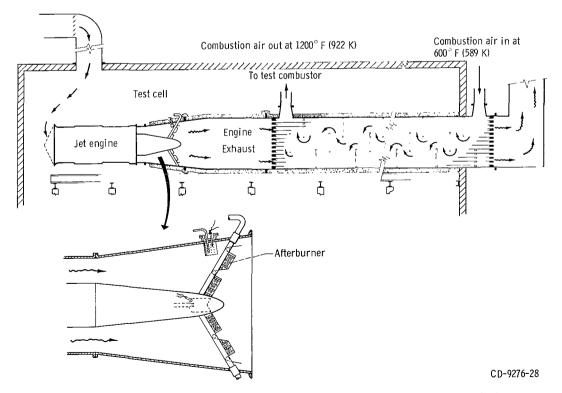


Figure 20. - Heat exchanger in exhaust of jet engine to provide high-temperature inlet air for test combustor.

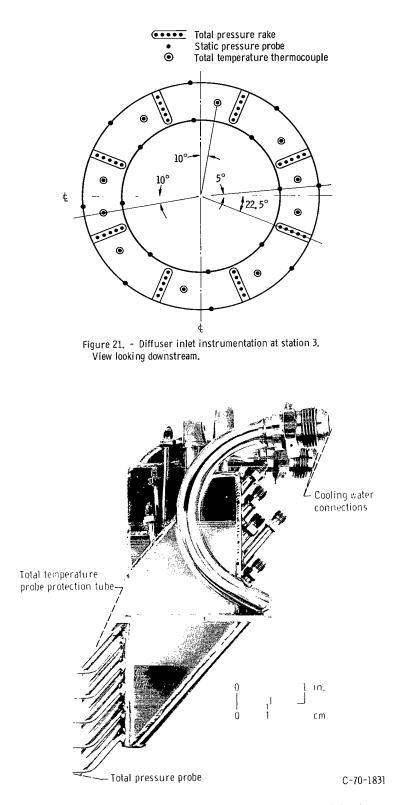


Figure 22. - Five-point total temperature and total pressure water-cooled probe assembly.

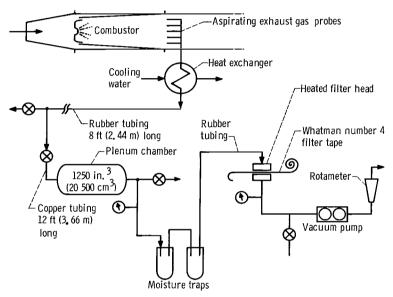


Figure 23. - Schematic of apparatus used to obtain smoke number data.

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