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1 **Numerical and experimental investigation on the ballast flushing system**

2 **Han Yuan^{1*}, Peilin Zhou² and Ning Mei¹**

3 *1. College of Engineering, Ocean University of China, 238 Songling Road, Laoshan*
4 *district, Qingdao 266100, China*

5 *2. Department of Naval Architecture and Marine Engineering, University of*
6 *Strathclyde, Glasgow G4 0LZ, United Kingdom*

7
8 Phone/Fax: +86-532-66781105,

9 E-mail: hanyuan@ouc.edu.cn

10 Address: 238 Songling Road, Laoshan district, Qingdao 266100, China

11
12 **Abstract**

13 The ballast sediments deposit not only provide the breeding ground for the survival
14 organisms, but also affect the weight balance of the ship and even accelerate the
15 corrosion of the ballast tank. In this work the performance of a ballast water flushing
16 system for the 138,000 m³ LNGC (Liquefied Natural Gas Carrier) double bottom
17 cargo ship is studied. A simulation model of the ballast tank was made to conduct the
18 numerical analysis. Besides, a scaled experimental setup was established on basis of
19 the similarity principle. With different injecting velocities at the flushing inlet, the
20 sediments distribution in the ballast tank is investigated and the energy consumption
21 of the circulating pump is studied. The results show that by flushing the ballast water
22 on the bottom, the sediments first accumulate at the far end, with the sediments
23 volume fraction climbs up to 10-30%, before gradually getting removed over time.
24 Further, higher inlet velocity leads to a more rapid decrease of average sediments

25 proportion in the ballast tank over time, but the energy consumption in circulating
26 pump significantly increases as well. The required power for this proposed ballast
27 water flushing system is within the common range and thus applicable in the cargo
28 ship.

29 **Keywords:** Ballast water; sediments; flushing; CFD.

30

31 **1. Introduction**

32 Negative environmental impacts made by the uptake and discharge of ballast water
33 are great challenge for international shipping. The non-indigenous species (NIS)
34 transported in the ballast tank may cause harm to native ecosystems [1, 2] and these
35 invasive species can even contribute to animal extinctions in local area [3-5]. In
36 response to this challenge, the International Maritime Organization (IMO) instituted a
37 performance standard of “Regulation D-2” for ballast water treatment management
38 [6-8]. In this regulation, limitations on both the size and the quantity of the remained
39 organisms in the treated ballast water are made to exclude microbes and viable
40 microorganisms from discharged ballast water [9]. To meet the requirement of the
41 IMO convention, different sorts of ballast water management technologies including
42 oxidation by chlorine/ozone and the ultraviolet radiation (UV) method are proposed
43 by researchers.

44 Although dozens of shipboard treatment systems have been certified as meeting
45 ballast water discharge standards till now, their application in eliminating invasive
46 species is not that satisfactory[10, 11]. It should be noted that only the maximum
47 amount of living organisms is restricted according to Regulation D-2, nevertheless,

48 the remaining sediments in the ballast water is still not concerned. Studies in
49 references [12, 13] pointed out that the soil sediments mainly consist of the clay, silt
50 and sand, with the particle diameter from less than 2 μ m to 2mm. Moreover, these soil
51 sediments are admitted with ballast water during the ballasting. Although most of the
52 large-size organisms (>10 μ m) gets inactivated according to Regulation D-2, a tiny
53 fraction of organisms cannot be totally removed from the loaded ballast water, and
54 these soil sediments turn to be the perfect breeding ground for the survival organisms
55 [14-16].

56 In fact, providing the habitat for organisms is only part of negative impacts that the
57 ballast water sediments could induce. On one hand, it is found that The sediments at
58 the bottom of the ballast tanks in a double hull cargo vessel can accumulate up to 30
59 cm depth within only two years operation [17]. According to current Rules and
60 Regulations of respective Classification Societies, sediments in ballast water tank can
61 only be systematically removed during the mandatory dry docking, and the interval is
62 usually made in every five years [18]. With such a long period of time the sediments
63 tend to be compacted and the sediments removing work becomes a great challenge.
64 On the other hand, the ballast sediments also affect the weight balance of the ship.
65 Due to non-uniform distribution in the ballast tanks, the loading and unloading of
66 ballast have to be cautious, because the excessive stresses can potentially lead to a
67 ship breaking during incorrectly unloading [19].

68 Furthermore, the corrosion of the ballast tank is another negative impact induced by
69 the ballast sediments. The sulphate reducing bacteria (SRB) and acid producing
70 bacteria (APB) that living in the sediments can bring significant microbiologically
71 influenced corrosion (MIC) [20, 21]. Compared with the electrochemical oxidation
72 process, the corrosion speed of MIC is greatly accelerated and thus unpredictable.

73 This paper proposed a solution for this problem. With the flushing system, the
74 unfiltered sediments along with the regrown organisms can be removed when needed

75 In this paper, a ballast water flushing system is proposed. By circulating the ballast
76 water in the tank, the deposits of sediments can be suspended and removed before
77 getting compacted on the tank bottom. A simulation model of the ballast tank was
78 made; an experimental setup was established on basis of the similarity principle.
79 Further, with four different ejecting velocities, both numerical and experimental
80 studies were conducted to evaluate the performance of this flushing system.

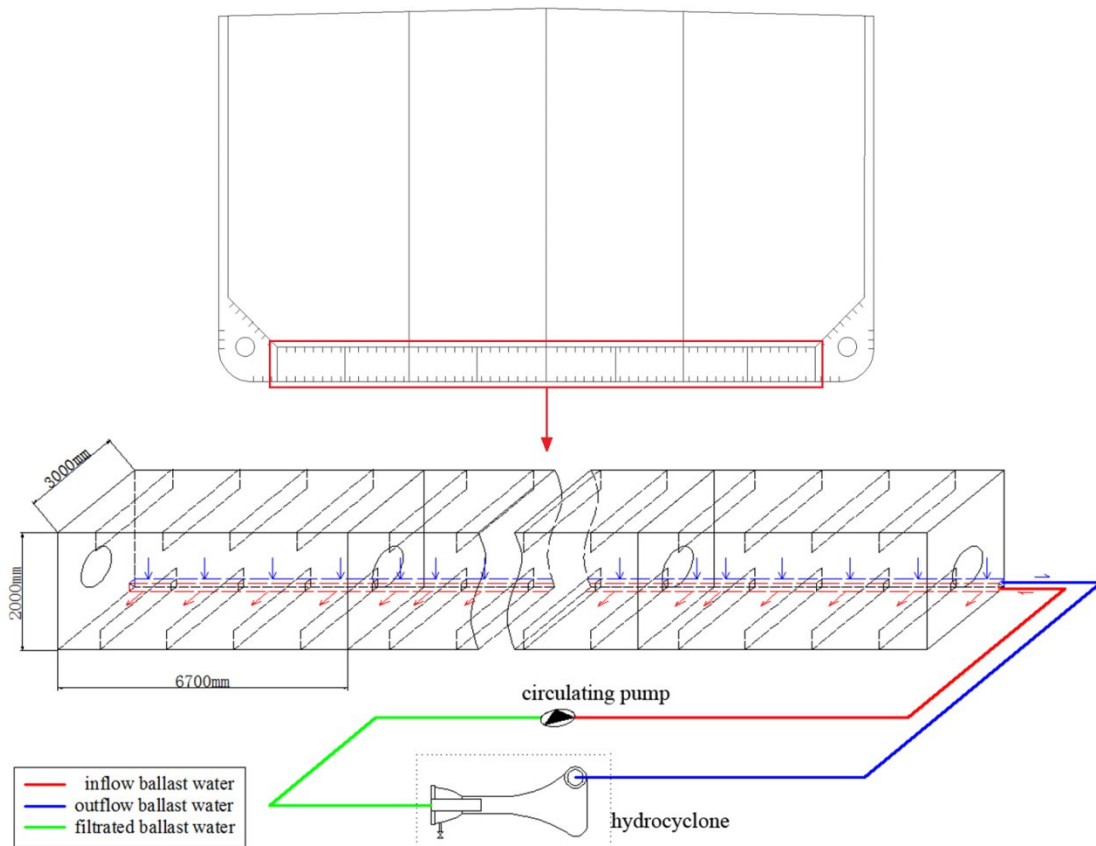
81

82 **2. System description**

83 In this work the ballast water system of the 138,000 m³ LNGC (Liquefied Natural Gas
84 Carrier) double bottom cargo ship [22] is investigated. The volume of cargo ballast
85 water system takes up to 56,090 m³ in total. Figure 1 shows the structure of a single
86 block of ballast tank in the cargo ship. As is shown, lines of longitudinal are arranged
87 on the inner side of the ballast tank and drain holes are provided on the bottom
88 longitudinal. Besides, the ballast water pipes (inlet-pipe and outlet-pipe) are arranged
89 in the corner, on which a row of jet holes (inlets) are placed along the inflow-pipe
90 while a row of exit holes (outlets) are placed along the outflow-pipe. Besides, a
91 hydrocyclone [23] is introduced between the outflow-pipe and the circulating pump to
92 separate the sediments from the ballast water. The size of a single block of ballast
93 tank is 6700×3000×2000(*mm*), and the size of inlets/outlets on the ballast water
94 pipes is 100×10(*mm*).

95 With the assistance of the circulating pump, the ballast water is first pumped into the
96 inflow-pipe and then ejected through the inlets, and in this way the deposits of
97 sediments are stirred up. Further, the suspended sediments, along with the ballast
98 water, get sucked into the outflow-pipe through the outlets. Then the mixture of water
99 and sediments is separated in the hydrocyclone before pumped back into the ballast
100 tank. As ballast water flows into the hydrocyclone, a cyclonic flow is produced and
101 the centrifugal force drives the sediments toward the outer wall, so the clean water
102 can flow through the centre of the hydrocyclone into the circulating pump. Thus the
103 ballast water circulates around the ballast tank and the bottom sediments deposits are
104 suspended and then removed from the ballast water.

105



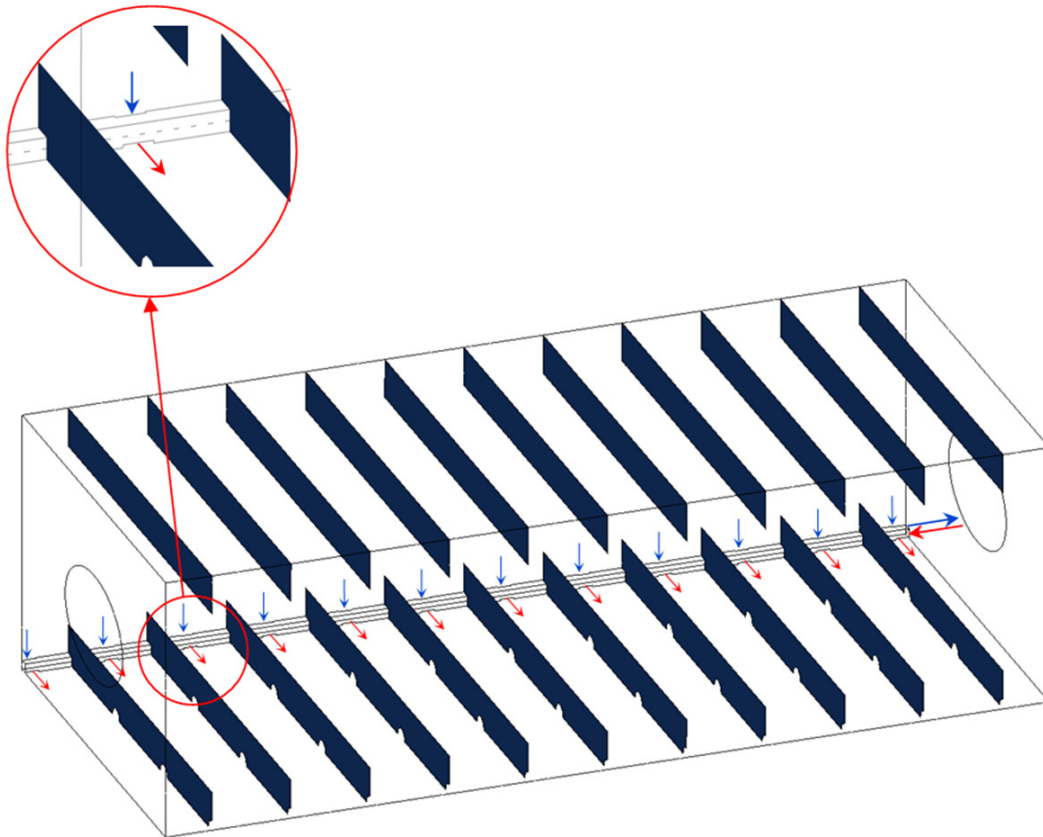
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107

Figure 1 Schematic of ballast water flushing system

108 **3. Simulation setup**

109 The numerical model for the ballast tank consists of a row of inlets along the inflow-
110 pipe, a row of outlets along the outflow-pipe and longitudinal with drain holes on the
111 inner side of the tank, as shown in Figure 2.



112

113 Figure 2 Ballast tank model for simulation

114

115 **3.1. Modelling equations**

116 The mixture model, a kind of Euler-Euler multiphase model, is used in this simulation.
117 The two phases of water and sediment are treated as interpenetrating continua in the
118 mixture model. Besides, the momentum and continuity equations of the mixture and
119 the volume fraction equations of the secondary phase are solved. Compared with a

120 full Eulerian multiphase model, the mixture model is simpler, and more suitable for
 121 the cases that the interphase laws are unknown. Simultaneously, the RNG $k-\varepsilon$ model
 122 is selected as the turbulence model for the simulation. The RNG (Renormalization
 123 Group) $k-\varepsilon$ model is a refined Standard $k-\varepsilon$ model, which is also derived from the
 124 instantaneous Navier-Stokes equations and follows the $k-\varepsilon$ (turbulent kinetic energy
 125 and dissipation rate) two equations turbulence modelling framework. But compared
 126 with the Standard $k-\varepsilon$ model, the RNG $k-\varepsilon$ model introduces more terms of
 127 dissipation in transport equation, which make it more accurate for rapidly strained
 128 flows and swirling flows. Besides, this model accounts for low-Reynolds-number
 129 effects and provides not constant values but an analytical formula for turbulent
 130 Prandtl numbers. All of these refinements make the RNG $k-\varepsilon$ model more
 131 appropriate for current simulation.

132

133 Governing equations:

134 The continuity equation is given as follows:

$$135 \quad \frac{\partial}{\partial t}(\rho_m) + \frac{\partial}{\partial x_i}(\rho_m \bar{u}_m^i) = 0 \quad (1)$$

136 The momentum equation for the mixture is given as follows:

$$137 \quad \frac{\partial}{\partial t}(\rho_m \bar{u}_m^i) + \frac{\partial}{\partial x_j}(\rho_m \bar{u}_m^i \bar{u}_m^j) = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j}[\mu_m(\frac{\partial}{\partial x_i} \bar{u}_m^j + \frac{\partial}{\partial x_j} \bar{u}_m^i)] + \rho_m \bar{g}^i - \frac{\partial}{\partial x_j}[\rho_m \overline{u_i' u_j'}] \quad (2)$$

138 Where \bar{u}_m^i is the mass-averaged velocity, ρ_m is the mixture density, α_s and α_f are the
 139 volume fraction of water phase and sediment phase, respectively, μ_m is the viscosity

140 of the mixture, u_i' is the turbulent velocity fluctuation. These terms can be written as
 141 follows:

$$142 \quad \bar{u}_m^r = \frac{\sum_{k=1}^n \alpha_k \rho_k \bar{u}_k^r}{\rho_m} = \frac{\alpha_s \rho_s \bar{u}_s^r + \alpha_f \rho_f \bar{u}_f^r}{\rho_m} \quad (3)$$

$$143 \quad \rho_m = \alpha_s \rho_s + \alpha_f \rho_f \quad (4)$$

$$144 \quad \mu_m = \alpha_s \mu_s + \alpha_f \mu_f \quad (5)$$

$$145 \quad \alpha_s + \alpha_f = 1 \quad (6)$$

146 Moreover, the Reynolds-stress tensor $-\rho_m \overline{u_i' u_j'}$ can be written as:

$$147 \quad -\rho_m \overline{u_i' u_j'} = -\frac{2}{3} \rho_m k \delta_{ij} + \mu_t \left(\frac{\partial}{\partial x_i} \bar{u}_m^r + \frac{\partial}{\partial x_j} \bar{u}_m^r \right) \quad (7)$$

$$148 \quad \mu_t = C_\mu \rho_m \frac{k^2}{\varepsilon} \quad (8)$$

149 Where k is the turbulent kinetic energy, ε is the turbulent dissipation rate, δ_{ij} is the
 150 Kronecker delta. μ_t is the eddy viscosity, C_μ is a constant.

151 In addition, the momentum equation for the mixture is not closed unless the
 152 Reynolds-stress tensor term is provided. This term can be obtained with assistance of
 153 the turbulence model shown as follow.

154 The transport equations for the RNG $k - \varepsilon$ model can be described as follows:

$$155 \quad \frac{\partial}{\partial t} (\rho k) + \frac{\partial}{\partial x_j} (\rho k u_j^r) = \frac{\partial}{\partial x_i} (\alpha_k \mu_{eff} \frac{\partial}{\partial x_i} k) + G_k + G_b - \rho \varepsilon \quad (9)$$

$$156 \quad \frac{\partial}{\partial t}(\rho\varepsilon) + \frac{\partial}{\partial x_j}(\rho\varepsilon u_j) = \frac{\partial}{\partial x_i}(\alpha_\varepsilon \mu_{eff} \frac{\partial}{\partial x_i} \varepsilon) + C_{1\varepsilon} \frac{\varepsilon}{k} (G_k + C_{3\varepsilon} G_b) - C_{2\varepsilon} \rho \frac{\varepsilon^2}{k} - R_\varepsilon \quad (10)$$

157 Where

$$158 \quad \mu_{eff} = \mu + \mu_t \quad (11)$$

$$159 \quad G_k = \mu_t S^2 \quad (12)$$

$$160 \quad S = \sqrt{2S_{ij}S_{ij}} \quad (13)$$

$$161 \quad S_{ij} = \frac{1}{2} \left(\frac{\partial}{\partial x_i} u_j + \frac{\partial}{\partial x_j} u_i \right) \quad (14)$$

162 In the $k-\varepsilon$ equations, G_k is the generation of turbulence kinetic energy due to the
 163 mean velocity gradients, G_b is the generation of turbulence kinetic energy due to
 164 buoyancy, which is neglected in this simulation, α_k and α_ε are the inverse effective
 165 Prandtl numbers for k and ε , μ_{eff} is the effective viscosity, $C_{1\varepsilon}$, $C_{2\varepsilon}$ and $C_{3\varepsilon}$ are
 166 turbulence model constants.

167 R_ε is the effects of rapid strain and streamline curvature, which reflects the main
 168 difference between the RNG and standard $k-\varepsilon$ model. It can be written as:

$$169 \quad R_\varepsilon = \frac{C_\mu \eta^3 (1 - \eta / \eta_0)}{1 + \beta \eta^3} \frac{\varepsilon^2}{k} \quad (15)$$

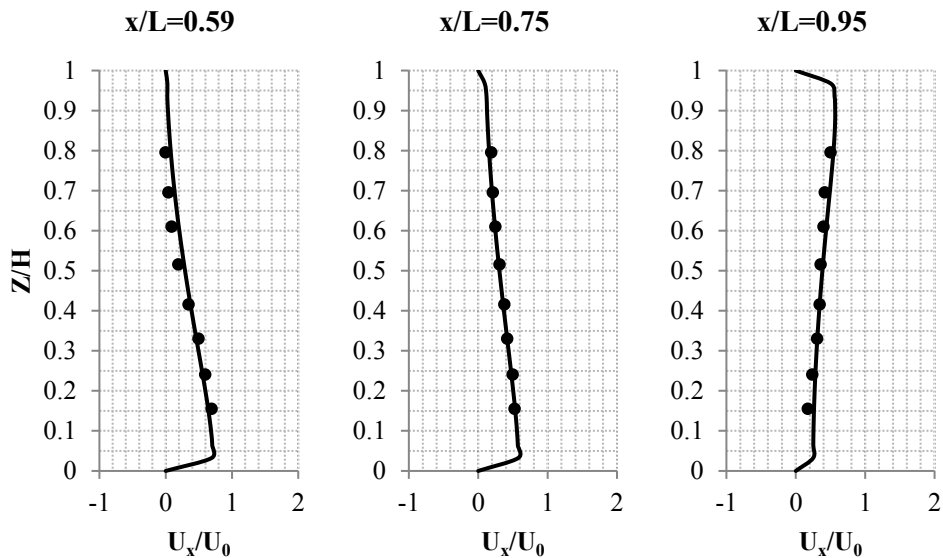
170 Where

$$171 \quad \eta = \frac{k}{\varepsilon} S \quad (16)$$

172 In the simulation, the values of above constants are taken as: $\alpha_k = \alpha_\varepsilon = 1.39$, $\beta = 0.012$,
 173 $\eta_0 = 4.38$, $C_{1\varepsilon} = 1.42$, $C_{2\varepsilon} = 1.68$, $C_{3\varepsilon} = 0$, $C_\mu = 0.0845$.

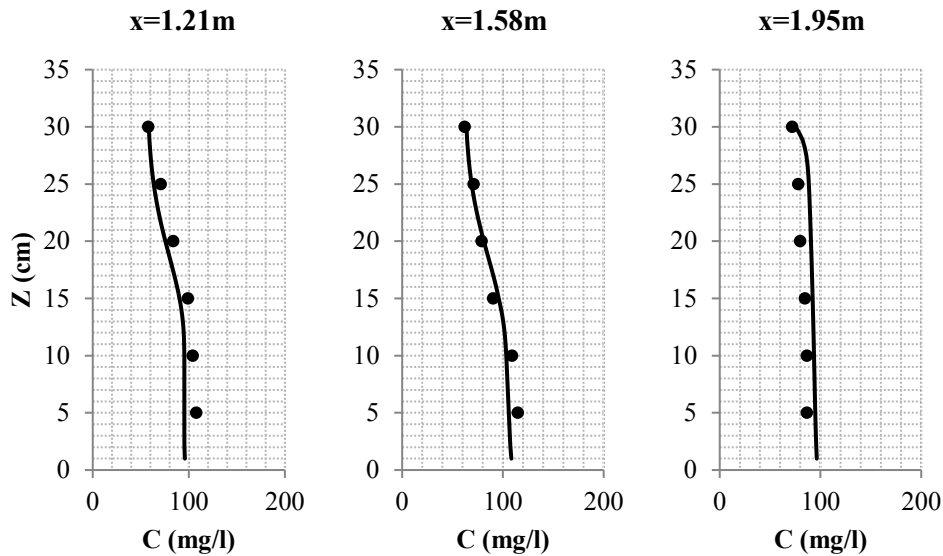
174 **3.2. Simulation method validation**

175 The sedimentation of sediments in the tank is a classic problem, which has been
176 extensively studied [24-28]. Therefore in this section, the reliability of above
177 mathematic model for sedimentation simulation is validated by selecting the
178 sedimentation tank in Ref. [24] as the research object. The geometry of the
179 sedimentation tank for the simulation is based on the experiment. The tank is a
180 rectangular one, with the length of 200 cm, width of 50cm and height of 31cm. The
181 inlet height is 10cm and the weir height is 30cm. The flow field and sediment
182 concentration of the sedimentation tank is simulated by utilizing the RNG $k-\varepsilon$ -
183 mixture model in this paper. The boundary condition and initial condition in this
184 simulation is assigned the same as that in the experimental research in Ref. [24]. The
185 simulation results are compared with the experimental results, shown in Figure 3. It is
186 found that the numerical simulation results show a good match with the experimental
187 results. Both the flow field and the sediment concentration distribution of the
188 sedimentation tank are well predicted, which indicates that the turbulence model and
189 multiphase model utilized in this study is adaptable for the sedimentation simulation.



191

(a)



192

193

(b)

194

- Simulation result with proposed mathematic model
- Experimental result in Ref. [20]

195 Figure 3 Comparison of simulation and experimental results: (a) dimensionless x -
196 velocity of suspended sediments; (b) vertical distribution concentration of suspended
197 sediments at different locations

198 3.3. Initial conditions and boundary conditions

199 It is known that the thickness of accumulated sediment in the ballast tank varies from
200 a few millimetres to several centimetres, but in most tanks is less than 50mm [29, 30].

201 Meanwhile, according to A. Tamburini [31, 32], the initial sediments volume fraction
202 on the bottom is experimentally measured at approximately 60%, so in this simulation,

203 the thickness of accumulated sediments on the vessel bottom is assumed to be 20mm

204 and the initial sediments volume fraction is made at 60%. Moreover, based on the

205 experimental data, the measured bulk density of sediments is made at 1500 kg/m^3 and

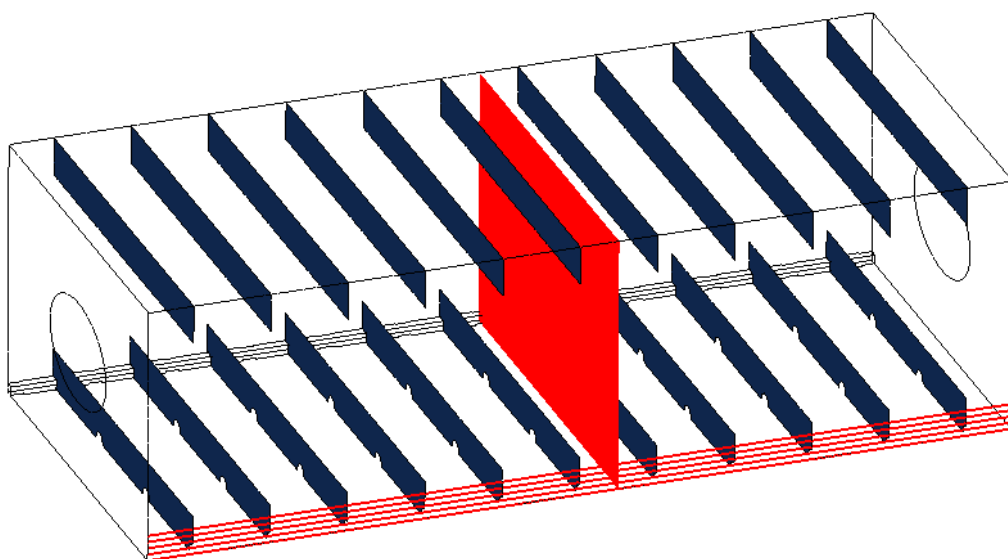
206 the median particle diameter is $11.64 \mu\text{m}$. The density of seawater is 1025 kg/m^3 .

207 Besides, the magnitude of the gravitational acceleration is 9.8 m/s^2 . Further, in the

208 simulation, the velocity-inlet boundary condition is assigned for the inlets on the
209 inflow-pipe. The magnitude of the inlet velocities are assigned at 4 constant values (at
210 5m/s, 10m/s, 15m/s, 20m/s, respectively) and the velocity direction is normal to the
211 inlet face. Besides, the pressure-outlet boundary condition is applied for the outlets on
212 the outflow-pipe.

213 **3.4. Monitoring plane and lines for simulation model**

214 A monitoring plane is used to obtain the sediments distribution of the ballast water.
215 Since the sediments are flushed and suspended by the ejected water along the ballast
216 tank bottom from left corner to right corner, the right corner sediments would be the
217 latest removed. Thus, 5 monitoring lines locate in the bottom right corner are used to
218 evaluate the suspending condition of the ballast tank. The location of these 5
219 monitoring lines are assigned at $z=0\text{mm}$, 10mm, 20mm, 30mm and 40mm, among
220 which lines of $z=0\text{mm}$, 10mm and 20mm are located in the initial sediments layer and
221 lines of $z=30\text{mm}$ and 40mm are located above the initial sediments layer. These
222 monitoring plane and lines are shown in Figure 4.



223

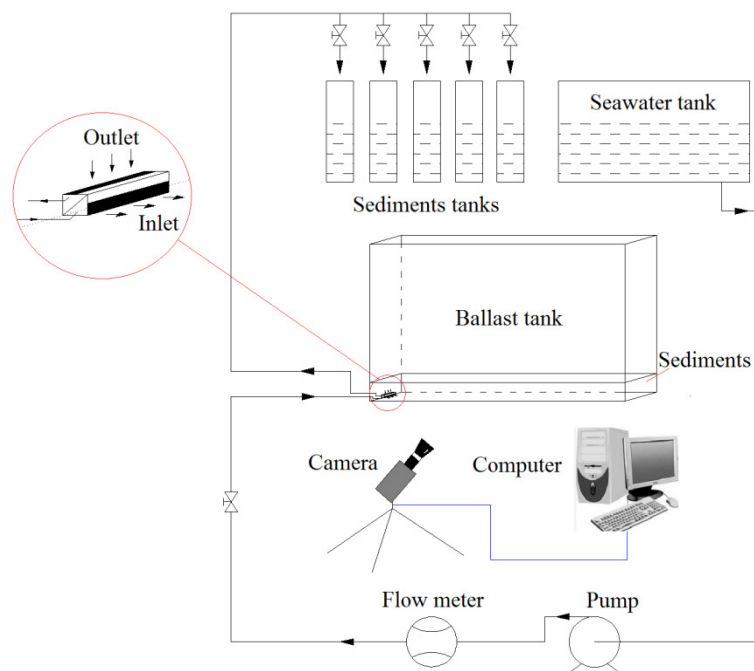
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Figure 4 Monitoring plane and lines for simulation results analysis

225 **4. Experimental setup**

226 **4.1. Test bench description**

227 A 1:10 scaled model of an acrylic ballast tank unit along with the test system was
228 constructed, shown in Figure 5. The bottom sediment in the ballast tank is
229 continuously flushed by the seawater that pumped from the seawater tank. The
230 outflow ballast water is then collected in the sediments tanks. In the experiments, the
231 injecting velocity of seawater at the inlet is controlled by the volume flow rate that
232 monitored with a flow meter. Further, a camera is used to monitor the sediments
233 distribution in the ballast tank.



234

235 **Figure 5 Experimental system**

236 The real-time measurement of the sediments proportion that remains in the ballast
237 tank is difficult. A. Tamburini [33] provided a method of quantifying the sediments
238 proportion in the entire tank. By introducing an impeller and measuring the density
239 caused pressure variation, the percentage of suspended solids can be calculated.

240 However, the impact of stirring process on the pressure of tank bottom is neglected in
241 this method, which leads to a low precision in the measurement.

242 In this study, the mean sediments proportion in the ballast tank is calculated based on
243 the weight of remained sediments in the ballast tank, which is ultimately measured by
244 the weight of sediments in the sediments tanks. The removed sediments are first
245 collected in the sediments tanks, and then weighted by oven drying method: after
246 being dried in the oven for 12 hours, the sediments are cooled to ambient temperature
247 in the dryer and weighted by an analytical balance. It is noted that a test consists of 3
248 replicates to arrive at the final result.

249 **Table 1 Parameters of instruments**

Analytical balance	0~0.2 kg, ±0.05%
Flowmeter	0.10~10 m ³ /h, ±1.5%

250

251 **4.2. Experimental plan**

252 In order to obtain a similar sediment removing process, the Froude number, which
253 represents the ratio of the flow inertia to the external field, is selected as the key
254 dimensionless number in the experiments[19], and the experimental parameters of
255 both the injecting velocity at the inlets and the required sediments removing time are
256 fixed based on the similarity principle. The Froude number is defined as:

$$257 \quad Fr = \frac{u}{\sqrt{gl}} \quad (17)$$

258 Where, u is the characteristic velocity of the flow, l is the characteristic length scale
259 of the flow.

260 Based on the similarity principle, the velocity scale and time scale for the
261 experimental model can be determined as follows:

$$262 \quad u_r = T_r = \sqrt{l_r} \quad (18)$$

263 Where, l_r is the geometric scale of the experimental model, fixed at 1:10.

264 In the tests, the thickness of accumulated sediments on the tank bottom is 2mm and
265 the initial sediments volume fraction is made at 60%. The bulk density of sediments is
266 made at 1500 kg/m³ and the measured median particle diameter is 11.64μm.
267 According to the calculation, the initial weight of the accumulated sediments on the
268 tank bottom is 328.86g. Before conducting the experiments, these sediments are first
269 fully suspended in the entire tank and then deposited in the tank, this process is to
270 make sure they are uniformly distributed after settling down on the bottom.

271 The experimental plan is listed in Table 2.

272 **Table 2 Experimental plan**

Experimental conditions	Inlet velocity m/s	Flow rate in pump L/s
Test 1	1.58	0.0158
Test 2	3.16	0.0316
Test 3	4.74	0.0474
Test 4	6.32	0.0632

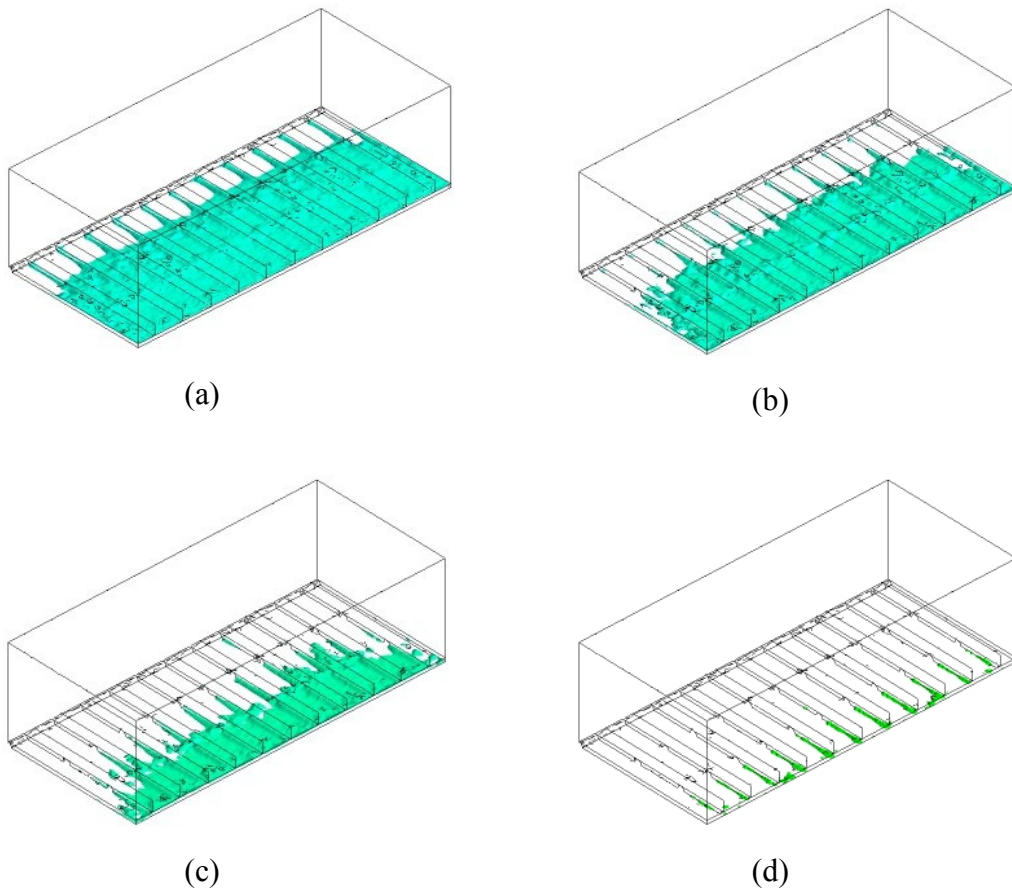
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274 **5. Results and discussions**

275 **5.1. Sediments distribution on the tank bottom**

276 Considering that the sediments discharged from the outlet of outflow-pipe are
277 separated in the hydrocyclone (with a relatively high separation efficiency of 98%),

278 thus, once the average proportion of the sediments in the ballast tank is less than 2%,
279 the sediments in the ballast tank can be considered adequately removed. In the
280 simulation, 4 inlet velocities of 5m/s, 10m/s, 15m/s and 20m/s are assigned. The
281 isosurfaces of sediments proportion above 2% at 600s are obtained and shown in
282 Figure 6.



283

284 **Figure 6 Isosurface of sediments proportion above 2% at 600s with different**
285 **inlet velocity: (a) inlet velocity at 5 m/s; (b) inlet velocity at 10 m/s; (c) inlet**
286 **velocity at 15 m/s; (d) inlet velocity at 20 m/s**

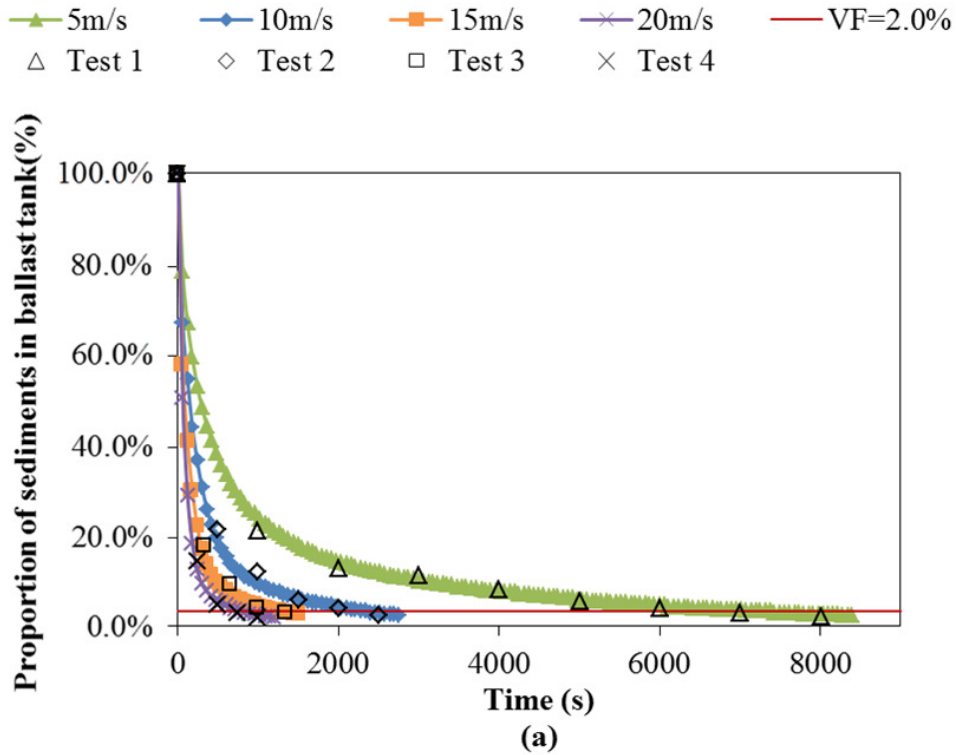
287 These results visually show the remaining bottom sediments in the tank and it is found
288 that the sediments are suspended from the bottom left. With a higher inlet velocity,

289 more sediment on the bottom is suspended at the same moment. Besides, since the jet
290 holes are located in the midline of two bottom longitudinal plates, the sediments along
291 this midline are first suspended. Also, it is found the sediments on the bottom right of
292 the tank are latest suspended, thus the sediments proportion on the bottom right of the
293 tank can be monitored, which indicates the suspension conditions of the ballast tank.
294 Based on this observed results, the monitoring lines on the bottom right of the ballast
295 tank are introduced in this study to evaluate the suspension conditions, and the results
296 are discussed in the following section.

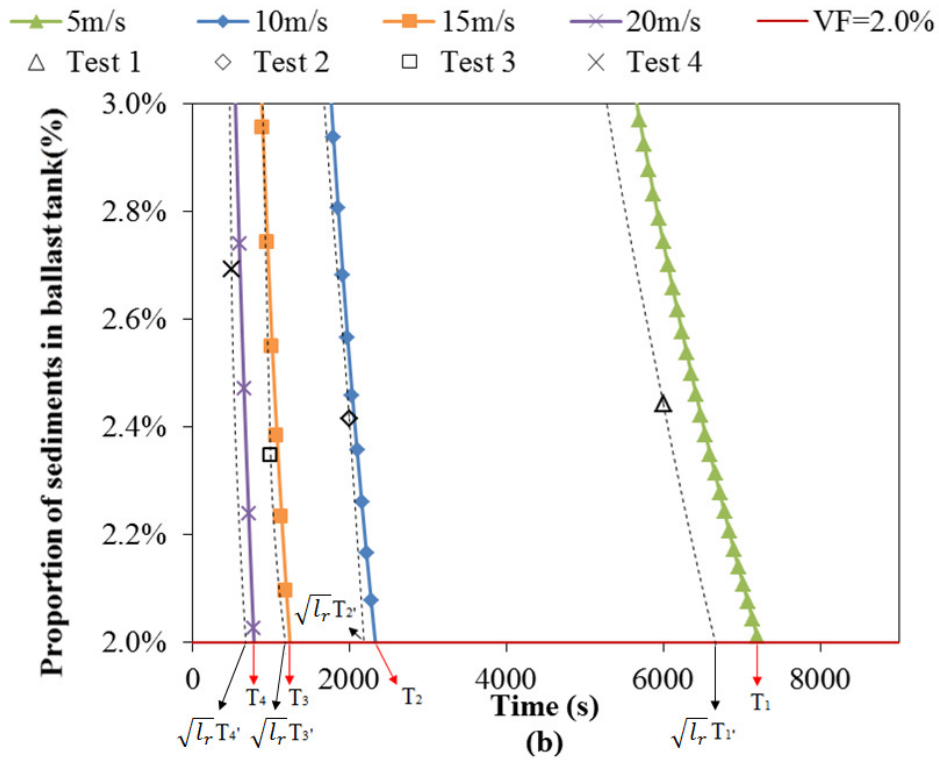
297 **5.2. Average sediments proportion in initial sediments layer**

298 The proportion of sediments remaining in the ballast tank over time is investigated in
299 this study, and the results are obtained and shown in Figure 7 (a).

300 Generally, the simulation results show that the average sediments proportion in the
301 ballast tank decreases over time, and higher inlet velocity leads to a more rapid
302 decrease. To compare the simulation results with the experimental one, the time scale
303 is taken into account, and these results show good match. Further, the decreasing
304 speed is quantitatively analysed by introducing the 2% proportion line. Figure 7 (b)
305 shows the results within a more narrow range in x-axis. To reduce the sediments
306 proportion to fewer than 2%, less required time is needed with higher inlet velocity.
307 For the simulation results, the required time with inlet velocity at 5m/s, 10m/s, 15m/s,
308 20m/s are approximately 7200s (T_1), 2300s (T_2), 1250s (T_3) and 790s (T_4),
309 respectively. In comparison, the experimental required time, after being processed on
310 basis of the time scale, is obtained. The results show that the required time
311 (considering the time scale) are approximately 6670s ($\sqrt{t_r} T_1'$), 2190s ($\sqrt{t_r} T_2'$), 1180s
312 ($\sqrt{t_r} T_3'$) and 680s ($\sqrt{t_r} T_4'$), respectively.



313



314

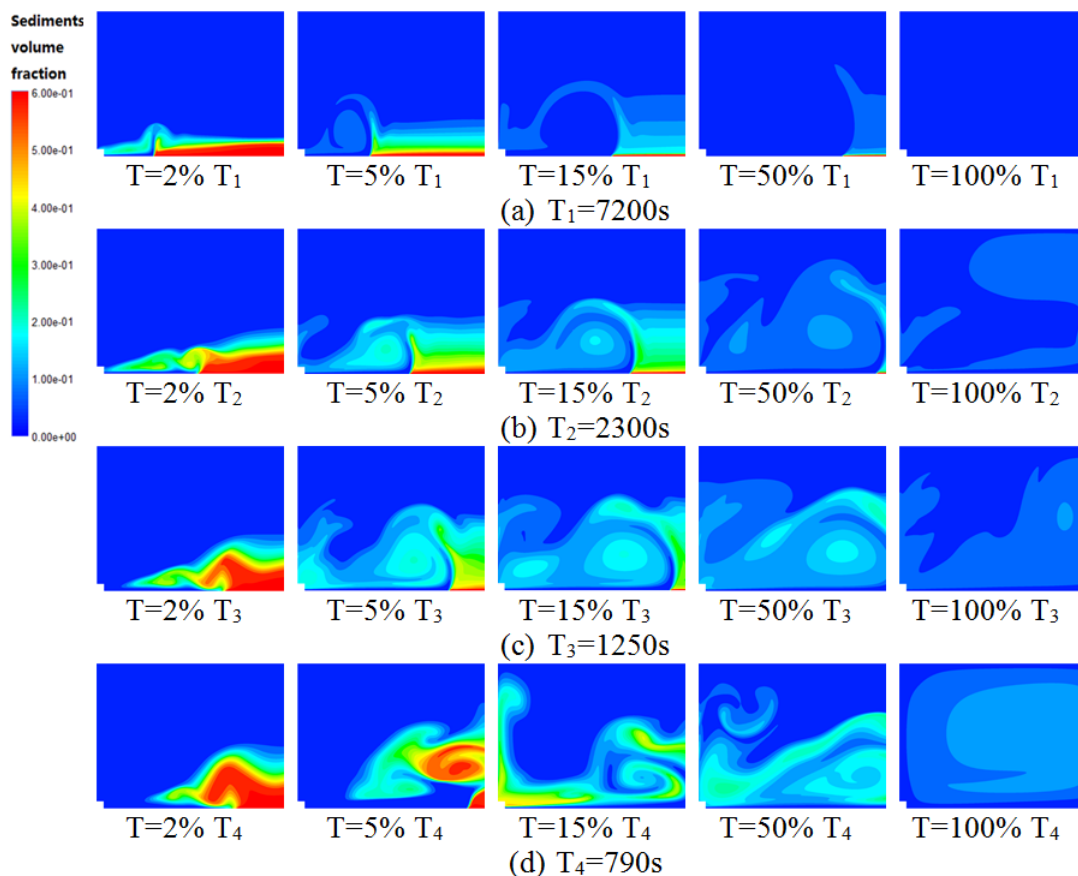
315 **Figure 7 Average sediments proportion in initial sediments layer at different**

316

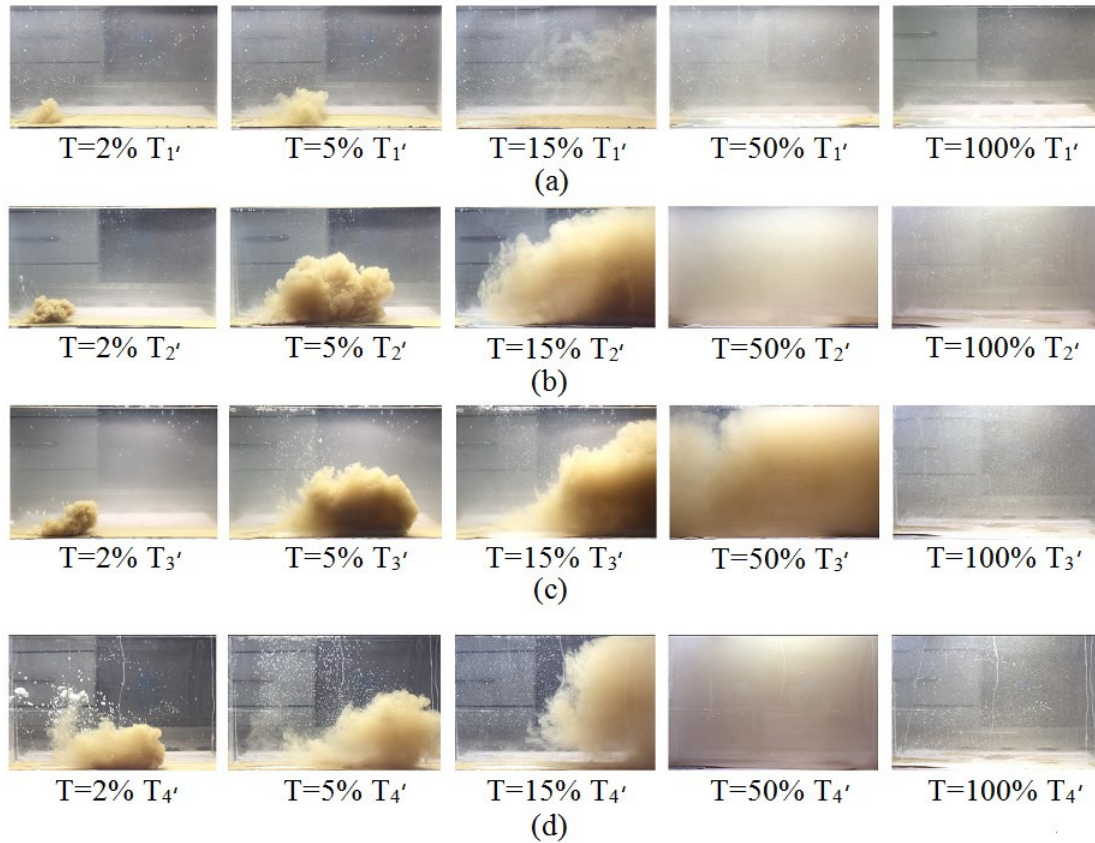
inlet velocities

317 **5.3. Sediments distribution on the monitoring plane**

318 Figure 8 shows the simulation results of sediments distribution on the monitoring
 319 plane as time varies. According to the simulation results in section 5.2, T_1 , T_2 , T_3 , T_4
 320 represent the required time of which the sediment are completely removed with 4
 321 different inlet velocities. Further, a comparison study is made by the experiments,
 322 shown in Figure 9. With the assistance of these studies, the flushing process inside the
 323 ballast tank is observed. Generally, the sediments on the ballast tank bottom are first
 324 stirred up by the water injected from the jet holes, and then be sucked into the exit
 325 holes and get removed, remaining very small part of the sediments suspending in the
 326 ballast tank.



328 Figure 8 Simulation results of sediments distribution on the monitoring plane with
 329 different inlet velocity: (a) inlet velocity at 5 m/s; (b) inlet velocity at 10 m/s; (c) inlet
 330 velocity at 15 m/s; (d) inlet velocity at 20 m/s



331
 332 Figure 9 Experimental results of sediments distribution with different inlet velocity: (a)
 333 inlet velocity at 1.58 m/s; (b) inlet velocity at 3.16 m/s; (c) inlet velocity at 4.74 m/s;
 334 (d) inlet velocity at 6.32 m/s

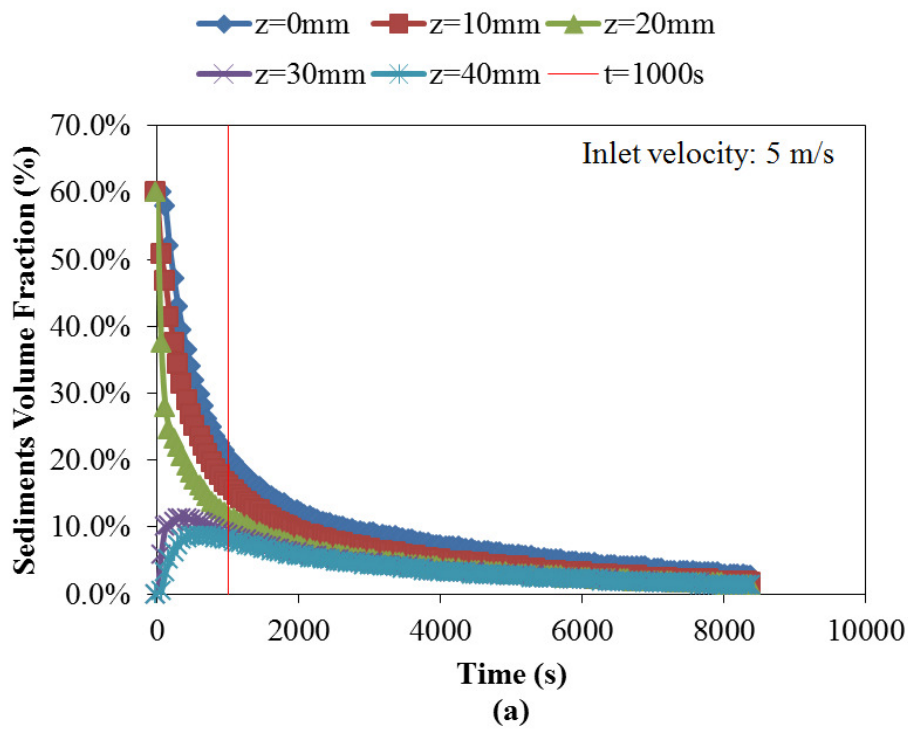
335 According to these results, the inlet velocity greatly affects the suspending of
 336 sediments. With a lower inlet velocity, the sediments are gradually stirred up from
 337 bottom left to right, while this process occurs drastically when the inlet velocity is at a
 338 higher level. The removing process is observed from this figure. The bottom
 339 sediments are suspended with the injected water, and then get discharged from the
 340 exit holes. Also, lower inlet velocity means that much longer time is needed to fully

341 suspend the bottom sediments. As is shown in Figure 8 (a), it takes more than 7000s
342 to fully remove the sediments with the inlet velocity at 5m/s. Compared with this,
343 Figure 8 (d) shows that only approximately 900s is needed to get a similar result with
344 the inlet velocity at 20m/s. And similar phenomenon can be observed in the
345 experimental study.

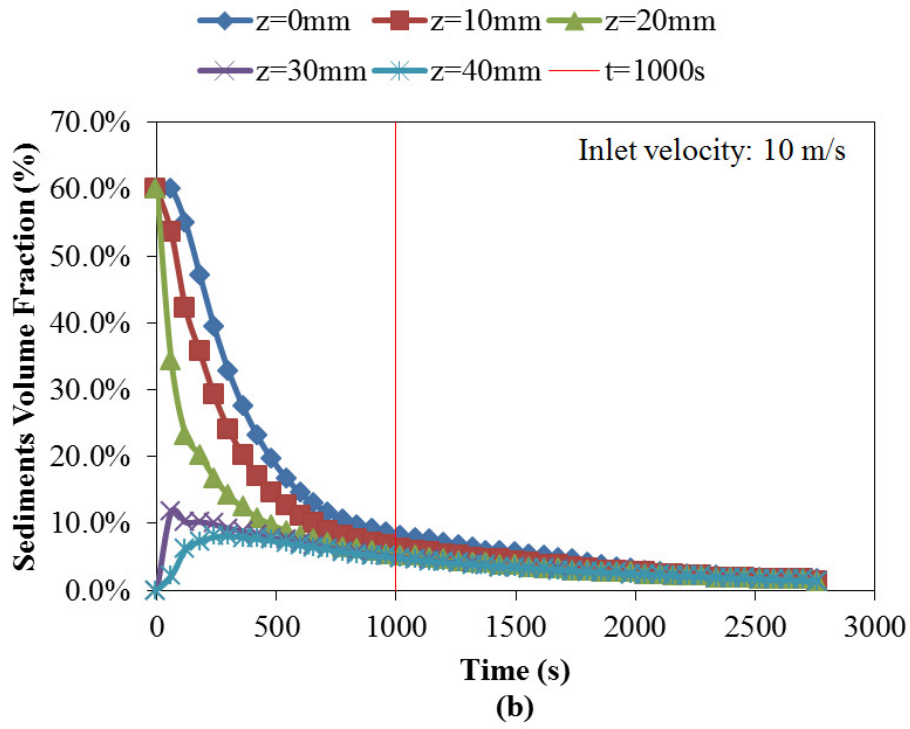
346 **5.4. Average sediments volume fraction on the monitoring lines**

347 According to the initial conditions made in this simulation, the average sediments
348 volume fractions on each monitoring line are obtained, shown in Figure 10. Similarly,
349 the flow time is also selected as the variable parameter. Noting that when $t=0s$, the
350 corresponding sediments volume fraction on different monitoring lines are at different
351 values. As it was assigned in the initial conditions, for monitoring lines of $z=0mm$,
352 10mm and 20mm, the sediments volume fraction are all 60%; while for monitoring
353 lines of $z=30mm$ and 40mm, this value are both at 0. As time varies, it is found that
354 the sediments volume fraction draw curves differently. On the monitoring lines of
355 $z=0mm$, 10mm and 20mm, where these lines are within the initial sediments layer, the
356 sediments volume fractions decrease rapidly from 60% to less than 20% over time,
357 and then they draw slightly decrease to 0. On the monitoring lines of $z=30mm$ and
358 40mm, it is noted that the sediments volume fractions first climb up to 10-30% before
359 decrease gradually to 0. This is attributed to the following reason: as the sediments
360 near the jet holes are stirred up, they are entrained by the injected water and
361 accumulated on the right bottom of the ballast tank firstly, and then these sediments
362 are removed slightly over time. This phenomenon has also been detected in Figure 8
363 and Figure 9.

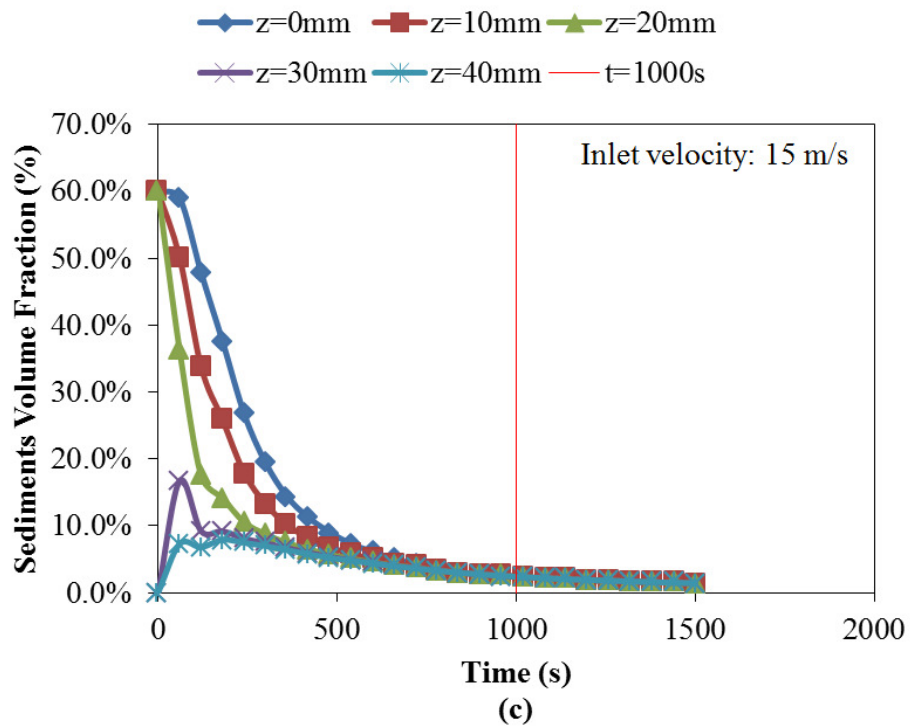
364 With different velocity at inflow-pipe inlet, the average sediments volume fractions of
 365 ballast water at the same moment are also quite different. When $t=1000s$, the
 366 sediments volume fractions on the tank bottom ($z=0mm$) are obtained. With inlet
 367 velocity at $5m/s$, $10m/s$, $15m/s$ and $20m/s$, this parameter decreases to approximately
 368 22.3% , 8.6% , 2.5% and 1.3% , respectively. This indicates that higher inlet velocity
 369 results in a better sediments suspension on the bottom. Besides of the tank bottom
 370 monitoring line, other monitoring lines are all found decrease drastically as the inlet
 371 velocity increases. Thus it is an effective method of removing the sediments by
 372 increasing the inlet velocity of inflow-pipe.



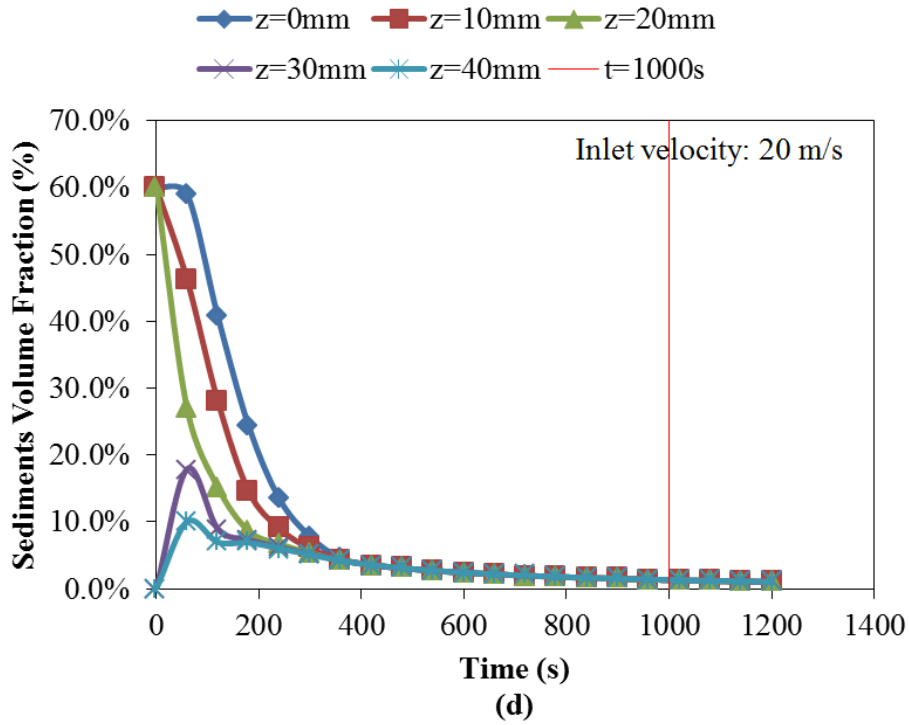
373



374



375



376

377 Figure 10 average sediments volume fraction on the monitoring lines

378 **5.5. Energy consumption of the pump at different inlet velocities as the**
 379 **sediments proportion in initial sediments layer varies**

380 Since the circulating pump is the only energy consumption component in this system,
 381 its energy consumption is the key parameter to evaluate the performance of this
 382 system. Generally, the energy consumption of the circulating pump is defined as:

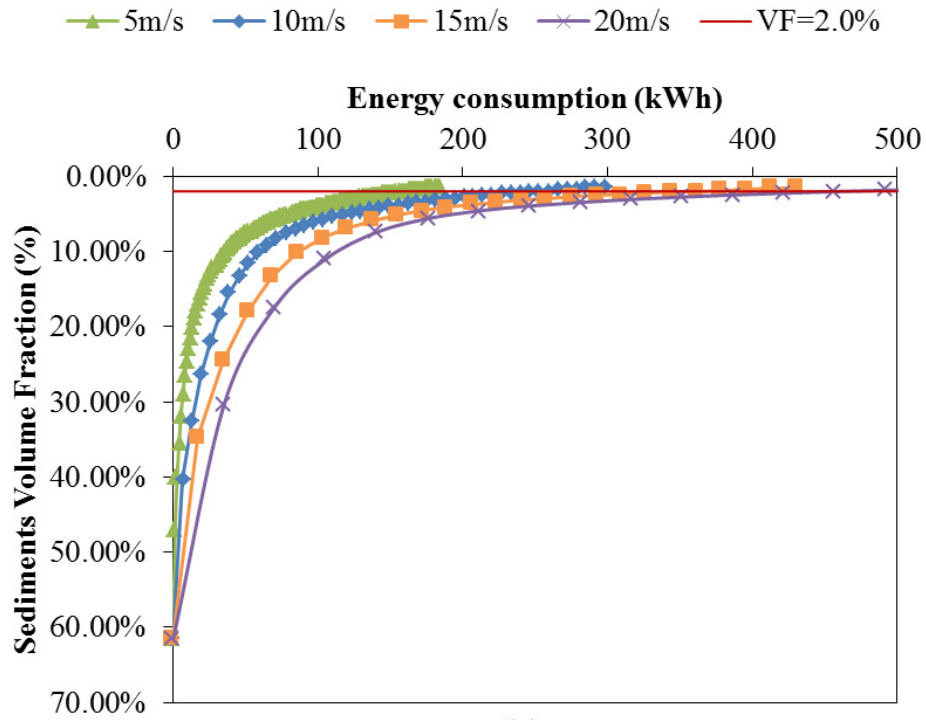
383
$$W_p = (E_{bw} + W_{loss})\eta_p \quad (19)$$

384
$$E_{bw} = (P_0 - P_1)\dot{m} / \rho_m \quad (20)$$

385
$$P_0 + \frac{1}{2} \rho_m u_0 = P_1 + \frac{1}{2} \rho_m u_1 \quad (21)$$

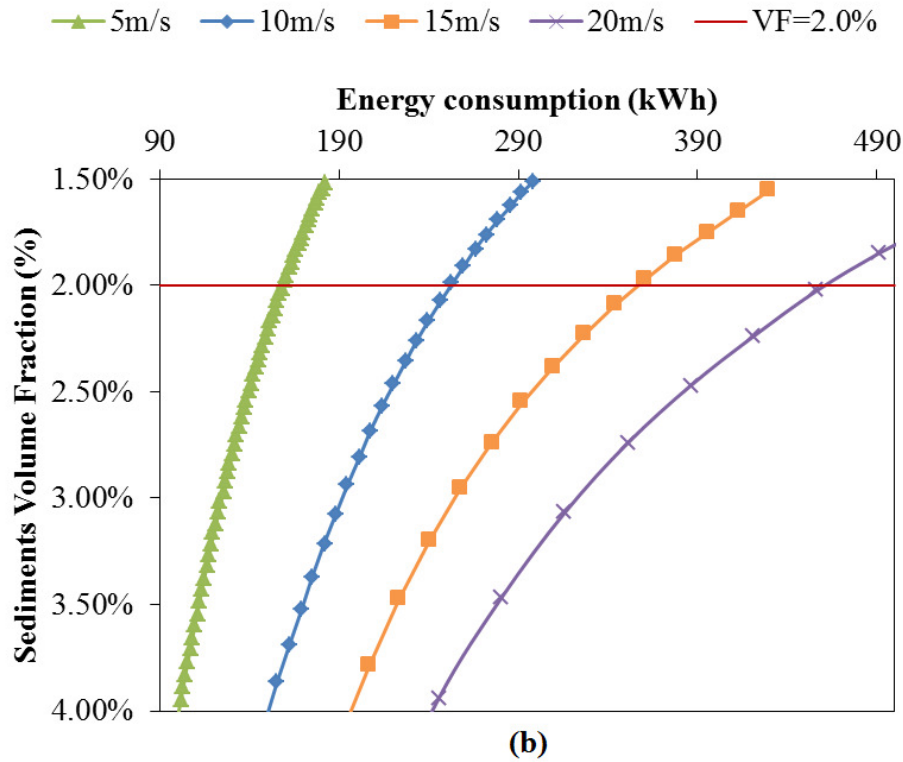
386
$$W_{loss} = \Delta P_{loss} \dot{m} / \rho_m \quad (22)$$

387 Where, E_{bw} is the total energy of the fluid in the ballast tank, P_0 and P_1 are the inlet
388 pressure of the inflow-pipe and the outlet pressure of the outflow-pipe, respectively,
389 W_{loss} is the energy losses of the pipe, ρ_m is the density of sediments-seawater mixture,
390 ΔP_{loss} is the pressure losses along the pipe.



391

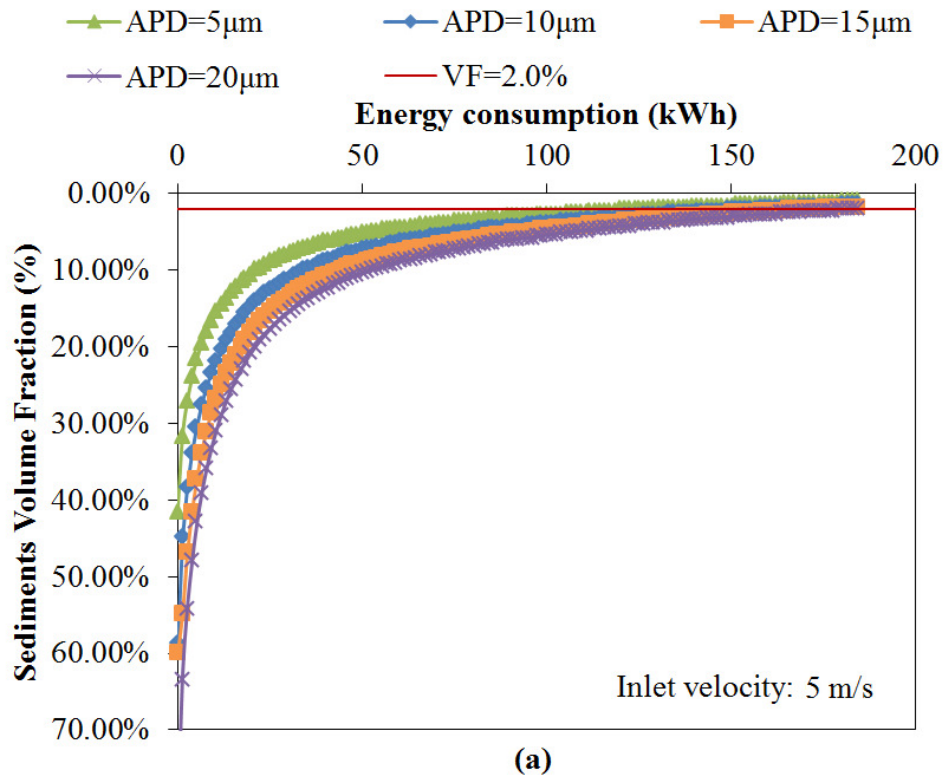
(a)



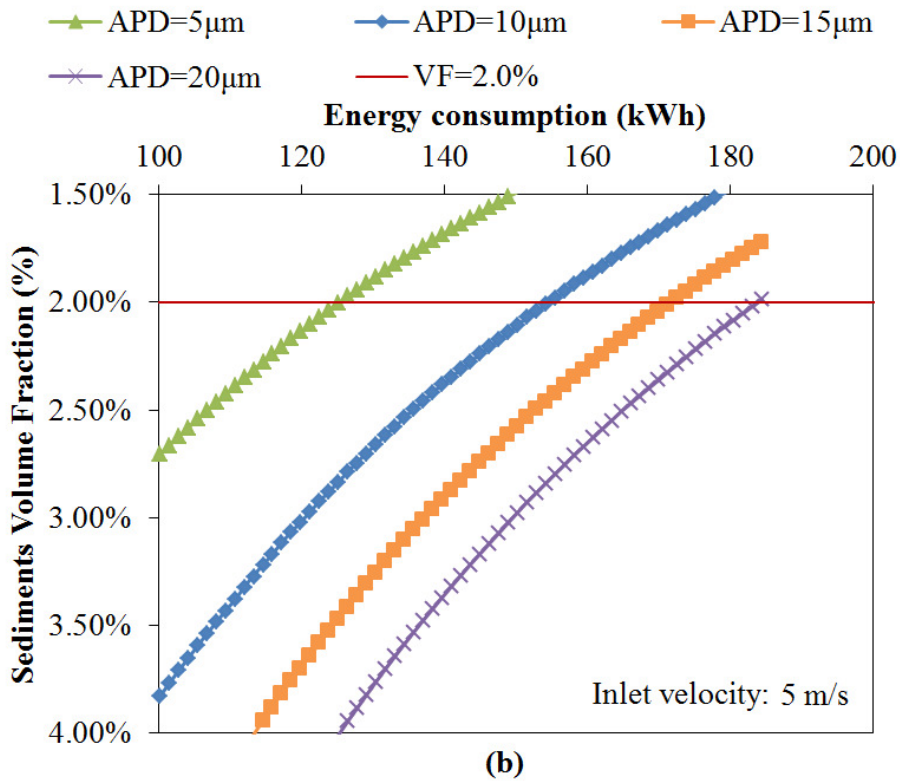
392

393 **Figure 11 Energy consumption of circulating pump at different inlet velocities**

394 According to Eq. (19-22), the energy consumption of the circulating pump is
 395 calculated and the results are shown in Figure 11. Generally, in order to get the same
 396 suspension effect, higher inlet velocity means higher energy consumption in
 397 circulating pump. The 2% sediments proportion line is also selected in this section as
 398 the critical line, which represents the fully suspension of the sediments. To reach this
 399 sediments proportion level, it is found that the energy consumption of 460kWh is
 400 required for inlet velocity at 20m/s. And this is followed by 356kWh, 252kWh and
 401 156kWh for inlet velocity at 15m/s, 10m/s and 5m/s. These results are reasonable, for
 402 higher inlet velocity in inflow-pipe leads to higher kinetic energy of inject water and
 403 higher resistance losses in pipes and tank. Therefore, higher inlet velocity can increase
 404 the sediments removing speed; nevertheless, it can also result in higher energy
 405 consumption in circulating pump.



406



407

408

Figure 12 Energy consumption of circulating pump as APD varies

409 Further, the effect of particle size on the performance of ballast water flushing system
410 is investigated. The energy consumption of the circulating pump with 4 different
411 average particle diameters (APD, at 5, 10, 15, 20 μ m, respectively) is compared. The
412 inlet velocity of this system is chosen at 5m/s. The simulation results are shown in
413 Figure 12. The result indicates that more energy will be consumed with larger particle
414 size. To reach the 2% sediments proportion level, it is found the energy consumption
415 of 125kWh is required for APD at 5 μ m. This is followed by 154kWh, 171kWh,
416 184kWh for APD at 10, 15 and 20 μ m.

417

418 **6. Conclusions**

419 This study proposed a ballast water flushing system for ballast sediments removing
420 purpose. By circulating the ballast water, the deposited sediments on the bottom of the
421 ballast tank can be stirred up and separated in the hydrocyclone. With the assistance
422 of this system, no modification is needed for the tank structure. Also, the evacuating
423 of ballast water is needless during the sediments removing process. With the flushing
424 system, the unfiltered sediments along with the regrown organisms can be removed
425 when needed. This proposed flushing system could be considered as an alternative
426 technique for ballast water management. In this study, the simulation model for this
427 flushing system is made. Moreover, a 1:10 scaled experimental setup is established
428 based on the similarity principle. Both simulation and experimental investigation are
429 conducted to reveal the performance of this flushing system. Based on the analysis
430 made in this study, conclusions can be drawn as follows:

- 431 • The inlet velocity significantly affects the sediments removing efficiency. With
432 higher magnitude of inlet velocity, the required time for suspending sediments on

433 the bottom of the tank can be greatly decreased.

- 434 • The sediments on the bottom of the ballast tank first accumulate at the far end,
435 with the sediments volume fraction climbs up to 10-30%. Then these sediments
436 get removed over time and the sediments volume fraction decreases gradually to
437 0.
- 438 • Higher inlet velocity leads to a more rapid decrease of average sediments
439 proportion in the ballast tank over time. For the simulation results, the required
440 time for totally removing of sediments is approximately 7200s (with inlet velocity
441 at 5m/s), 2300s (10m/s), 1250s (15m/s) and 790s (20m/s), respectively. Similar
442 results are obtained from the experiments.
- 443 • To get the same suspension effect, higher inlet velocity means much higher
444 energy consumption in circulating pump. The energy consumption of 460kWh is
445 required (with inlet velocity at 20m/s), followed by 356kWh, 252kWh and
446 156kWh with inlet velocity at 15m/s, 10m/s and 5m/s, respectively. Besides,
447 more energy will be consumed with larger particle size.

448

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