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Dynamic equations for a fully anisotropic piezoelectric rectangular plate

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1. Introduction

Piezoelectric materials have been used widely in applications for sensing and actuation purposes in recent years. As piezoelectric sensors and actuators usually are thin in comparison to relevant wavelengths, the analyzes of thin piezoelectric layers, such as beams, plates and shells, have attained considerable research interest. Many references to work on higher-order piezoelectric plate theories prior to 2000 are given in the review article by Wang and Yang [1]. Further references to laminated piezoelectric plates are presented in [2] while [3] presents a classification and comparison among higher-order piezoelectric plate models based on power series expansions. A more recent review article on three dimensional approaches for piezoelectric plates is presented by Wu et al. [4].

Plate theories for various material configurations were developed in the 50's by Mindlin, among which piezoelectric plates were addressed in [5]. This work was later generalized by Tiersten and Mindlin [6] and Mindlin [7] where two-dimensional equations for a piezoelectric plate were systematically derived using power series expansions for the mechanical and electric displacements. Bugdayci and Bogy [8] and Lee et al. [9,10] used trigonometric series expansions for piezoelectric plates, which provides an alternative approach for analyzing plate vibrations more suitable for high-

ABSTRACT

A hierarchy of dynamic plate equations based on the three dimensional piezoelectric theory is derived for a fully anisotropic piezoelectric rectangular plate. Using power series expansions results in sets of equations that may be truncated to arbitrary order, where each order set is hyperbolic, variationally consistent and asymptotically correct (to all studied orders). Numerical examples for eigenfrequencies and plots on mode shapes, electric potential and stress distributions curves are presented for orthotropic plate structures. The results illustrate that the present approach renders benchmark solutions provided higher order truncations are used, and act as engineering plate equations using low order truncation. © 2015 Elsevier Ltd. All rights reserved.

> frequency modes. More recently developed plate theories using various sorts of series expansions of the displacements and the electric potential for both single and laminated piezoelectric plates can be found in [11–24]. These expansions are either using a few power series terms, or written on a general higher order fashion that may be used to render solutions that converge to the three dimensional solutions. Exact three dimensional analyses for single and laminated piezoelectric plates having mixed (simply supported) boundary conditions are treated in [25–28]. Numerical methods such as finite element analysis (FEA) has been adopted on both classical and higher order series expansion theories [29–34].

> Recently Mauritsson et al. [35] have derived plate equations for a homogenous fully anisotropic elastic plate, using a systematic power series expansion approach, previously adopted for isotropic rods, beams, shells and plates [36–40]. The same general approach has been applied to various piezoelectric layer configurations [41-43]. In the present paper the work in [35] is extended to cover anisotropic piezoelectric plates. The method aims at systematically develop a hierarchy of equations for general piezoelectric plates using power series expansions in the thickness coordinate of the displacement components and the electric potential. Insertion of these expansions into the three dimensional equations of motion leads to recursion relations among the expansion functions, which can be used to eliminate all but some of the lowest order expansion functions. Hereby all fields can be expressed in a finite number of expansions functions without performing any truncations. The power series expansions are subsequently inserted into the three dimensional boundary conditions at the upper and the lower surfaces of the plate. These boundary conditions represent a set of eight scalar equations of motion, including eight unknown

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expansion functions, which constitute the system of plate equations. Using variational calculus, the pertinent edge boundary conditions for rectangular plates are obtained in an equally systematic manner. This hierarchy of piezoelectric plate equations can be truncated to any order in the thickness where each studied truncation order is asymptotically correct in line with [35,37,38,44].

The present approach generally differs in several respects from the cited works using power series expansion on piezoelectric plates. The main issues concern the derivation of exact recursion relations where only the lowest order expansion terms need to be considered. Another object is the procedure when collecting terms for the truncation process, resulting in variationally consistent equation systems that are asymptotically correct. It should also be noticed that the present equations are not confined to the static case. Moreover, the plate configuration may be of arbitrary anisotropy without any symmetry classes. One advantage with such a general analysis is that all other cases can be obtained as special cases. The previously derived plate equations for the fully anisotropic elastic case [35] can also be obtained as a special case. As a fully anisotropic, piezoelectric material is described by 21 independent stiffness constants, 18 independent piezoelectric coupling constants and six independent dielectric constants, the explicit expressions for the coefficients in the plate equations become very complicated. For this reason the plate equations for the most general case are derived in a very compact form as four matrix equations, including matrix operators which are recursively defined using the commercial code Mathematica.² Hereby it is straightforward to study all types of anisotropy configurations.

As the material configuration for a fully anisotropic material results in complicated expressions, it is natural to study simpler cases of orthotropic plates more in detail. Here the eight plate equations can be added and subtracted in pairs to obtain two uncoupled systems of equations, each of them including four equations and four unknowns. The two uncoupled systems correspond to the symmetric (in-plane) and antisymmetric (out-of-plane) part of the motion, respectively. These equations, including the edge boundary conditions, are explicitly given for the lower truncation orders. To validate the present plate equations results for dispersion curves, eigenfrequencies as well as potential, displacement and stress distribution curves are presented. Both single and laminated plates are studied and comparisons are made to other approximate theories as well as the exact three dimensional theory. The results illustrate both the benchmark property of the higher order truncations and the efficiency of the lower order engineering equations.

2. Problem formulation

Consider a homogeneous piezoelectric plate of thickness 2h according to Fig. 1. The material is fully anisotropic with density ρ . The basic equations governing the motion in a piezoelectric continuum are written with abbreviated subscripts [45] as

$$\nabla_{ij}T_J = \rho \partial_t^2 u_i, \tag{2.1}$$

$$\nabla_i D_i = 0. \tag{2.2}$$

Here vector subscripts are expressed through lower case letters i = x, y, z, while abbreviated subscripts are expressed through upper case letters I = 1, 2, 3, 4, 5, 6. The mechanical stress, mechanical displacement and electric displacement column matrices are defined through

$$[T_{I}] = (T_{xx}T_{yy}T_{zz}T_{yz}T_{xz}T_{xy})^{T}, \quad [u_{i}] = (u_{x}u_{y}u_{z})^{T}, \quad [D_{i}] = (D_{x}D_{y}D_{z})^{T}.$$
(2.3)



The divergence vector ∇_i is defined in the usual way, while the divergence stress operator ∇_{ij} is represented in matrix form through

$$[\nabla_{ij}] = \begin{pmatrix} \partial_x & \mathbf{0} & \mathbf{0} & \partial_z & \partial_y \\ \mathbf{0} & \partial_y & \mathbf{0} & \partial_z & \mathbf{0} & \partial_x \\ \mathbf{0} & \mathbf{0} & \partial_z & \partial_y & \partial_x & \mathbf{0} \end{pmatrix}, \quad [\nabla_{ij}] = [\nabla_{ij}]^T.$$
(2.4)

Partial derivatives are expressed as $\partial_x = \partial/\partial_x$ and so on.

For a linear elastic, piezoelectric material the constitutive equations that express the mechanical stresses and the electric displacements in terms of the mechanical displacements and the electric potential, are

$$T_I = c_{IJ} \nabla_{Jk} u_k + e_{Ij} \nabla_j \Phi, \tag{2.5}$$

$$D_i = e_{ij} \nabla_{jk} u_k - \epsilon_{ij} \nabla_j \Phi. \tag{2.6}$$

Here the quasistatic approximation is applied, i.e. the electric field is given as the gradient of the electric potential

$$E_i = -\nabla_i \Phi. \tag{2.7}$$

The various material parameters appearing in (2.5) and (2.6) are the 21 independent stiffness constants collected in the symmetric 6×6 matrix

$$[c_{ij}] = \begin{pmatrix} c_{11} & c_{12} & c_{13} & c_{14} & c_{15} & c_{16} \\ c_{12} & c_{22} & c_{23} & c_{24} & c_{25} & c_{26} \\ c_{13} & c_{23} & c_{33} & c_{34} & c_{35} & c_{36} \\ c_{14} & c_{24} & c_{34} & c_{44} & c_{45} & c_{46} \\ c_{15} & c_{25} & c_{35} & c_{45} & c_{55} & c_{56} \\ c_{16} & c_{26} & c_{36} & c_{46} & c_{56} & c_{66} \end{pmatrix},$$

$$(2.8)$$

the 6 independent dielectric constants collected in the symmetric 3×3 matrix

$$[\epsilon_{ij}] = \begin{pmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} \end{pmatrix},$$
(2.9)

and the 18 independent piezoelectric coupling constants collected in the 3×6 piezoelectric coupling matrix

$$[e_{ij}] = \begin{pmatrix} e_{x1} & e_{x2} & e_{x3} & e_{x4} & e_{x5} & e_{x6} \\ e_{y1} & e_{y2} & e_{y3} & e_{y4} & e_{y5} & e_{y6} \\ e_{z1} & e_{z2} & e_{z3} & e_{z4} & e_{z5} & e_{z6} \end{pmatrix}, \quad [e_{ij}] = [e_{ij}]^{T}.$$
 (2.10)

Insertion of (2.5) and (2.6) into (2.1) and (2.2) gives the governing equations for the displacements and the electric potential

$$\nabla_{ij} c_{jK} \nabla_{Kl} u_l + \nabla_{ij} e_{jk} \nabla_k \Phi = \rho \partial_t^2 u_l, \qquad (2.11)$$

$$\nabla_i e_{il} \nabla_{lk} u_k - \nabla_i \epsilon_{ii} \nabla_i \Phi = 0. \tag{2.12}$$

3. Power series expansions

To derive plate equations, the displacement components and the electric potential are expanded in power series in the thickness coordinate

$$u_i(x, y, z, t) = \sum_{n=0}^{\infty} z^n u_i^{(n)}(x, y, t),$$
(3.1)

² Registered trademark of Wolfram Research Inc.

$$\Phi(x, y, z, t) = \sum_{n=0}^{\infty} z^n \Phi^{(n)}(x, y, t).$$
(3.2)

Insertion of the series expansions (3.1) and (3.2) into the governing equations of motion (2.11) and (2.12) gives power series expansion equations in the thickness coordinate *z*. By collecting equal powers of z, the solution of the equation system for each power yields the recursion formulas

$$u_{i}^{(n+2)} = \frac{1}{(n+1)(n+2)} \Big[A_{ij}^{(0)} u_{j}^{(n)} + (n+1) B_{ij}^{(0)} u_{j}^{(n+1)} + F_{i}^{(0)} \Phi^{(n)} + (n+1) G_{i}^{(0)} \Phi^{(n+1)} \Big],$$
(3.3)

$$\Phi^{(n+2)} = \frac{1}{(n+1)(n+2)\epsilon_{zz}} \Big[a_i^{(0)} u_i^{(n)} + (n+1) b_i^{(0)} u_i^{(n+1)} + f^{(0)} \Phi^{(n)} + (n+1) g^{(0)} \Phi^{(n+1)} \Big],$$
(3.4)

.

where n = 0, 1, 2, ... The introduced operators are defined as

$$\begin{split} A_{im}^{(0)} &= M_{ij} \bigg[\delta_{jm} \rho \partial_t^2 - \overline{\nabla}_{jK} c_{KL} \overline{\nabla}_{Lm} - \frac{1}{\epsilon_{zz}} e_j \overline{\nabla}_k e_{kL} \overline{\nabla}_{Lm} \bigg], \\ B_{im}^{(0)} &= -M_{ij} \bigg[L_{jK} c_{KL} \overline{\nabla}_{Lm} + \overline{\nabla}_{jK} c_{KL} L_{Lm} + \frac{e_j}{\epsilon_{zz}} (l_k e_{kL} \overline{\nabla}_{Lm} + \overline{\nabla}_k e_{kL} L_{Lm}) \bigg], \\ F_i^{(0)} &= M_{ij} \bigg[\frac{1}{\epsilon_{zz}} e_j \overline{\nabla}_k \epsilon_{kl} \overline{\nabla}_l - \overline{\nabla}_{jK} e_{Kl} \overline{\nabla}_l \bigg], \\ G_i^{(0)} &= M_{ij} \bigg[\frac{e_j}{\epsilon_{zz}} (l_k \epsilon_{kl} \overline{\nabla}_l + \overline{\nabla}_k \epsilon_{kl} l_k) - L_{jK} e_{Kl} \overline{\nabla}_l - \overline{\nabla}_{jK} e_{Kl} l_l \bigg], \\ a_k^{(0)} &= e_i A_{ik}^{(0)} + \overline{\nabla}_i e_{ij} \overline{\nabla}_{jk}, \\ b_k^{(0)} &= e_i B_{ik}^{(0)} + l_i e_{ij} \overline{\nabla}_{jk} + \overline{\nabla}_i e_{ij} L_{jk}, \\ f^{(0)} &= e_i F_i^{(0)} - \overline{\nabla}_i \epsilon_{ij} \overline{\nabla}_j, \\ g^{(0)} &= e_i G_i^{(0)} - l_i \epsilon_{ij} \overline{\nabla}_j - \overline{\nabla}_i \epsilon_{ij} l_j, \end{split}$$

$$(3.5)$$

where δ_{ii} is the Kronecker delta. Here new material matrices are introduced as

$$\begin{aligned} |e_{i}| &= (e_{z5}e_{z4}e_{z3}), \\ [C_{ij}] &= \begin{pmatrix} c_{55} + e_{z5}^{2}/\epsilon_{zz} & c_{45} + e_{z4}e_{z5}/\epsilon_{zz} & c_{35} + e_{z3}e_{z5}/\epsilon_{zz} \\ c_{45} + e_{z4}e_{z5}/\epsilon_{zz} & c_{44} + e_{z4}^{2}/\epsilon_{zz} & c_{34} + e_{z3}e_{z4}/\epsilon_{zz} \\ c_{35} + e_{z3}e_{z5}/\epsilon_{zz} & c_{34} + e_{z3}e_{z4}/\epsilon_{zz} & c_{33} + e_{z3}e_{z4}/\epsilon_{zz} \end{pmatrix}, \end{aligned}$$
(3.6)

where $[M_{ij}] = [C_{ij}]^{-1}$. Hence, $[C_{ij}]$ can be interpreted as a matrix of stiffened elastic constants. Moreover the introduced ordering matrices are

$$[l_i] = (001), \quad [L_{ij}] = \begin{pmatrix} 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \end{pmatrix}, \quad [L_{ij}] = [L_{ij}]^T. \quad (3.7)$$

The operators with an overline $(\overline{\nabla}_i, \overline{\nabla}_{il} \text{ and } \overline{\nabla}_{li})$ are differential operators in the plate plane obtained through

$$\nabla_{i} = \overline{\nabla}_{i} + \partial_{z} l_{i}, \quad \nabla_{ij} = \overline{\nabla}_{ij} + \partial_{z} L_{ij}, \quad \nabla_{lj} = \overline{\nabla}_{lj} + \partial_{z} L_{lj}.$$
(3.8)

Consequently $\overline{\nabla}_{ij}$ and $\overline{\nabla}_{lj}$ are obtained by putting all derivatives with respect to z to zero in (2.4).

The recursion relations (3.3) and (3.4) can be used repeatedly to express all expansion functions in the lowest-order ones with an upper index n = 0, 1. The expansion functions can then be written as

$$u_i^{(n)} = \frac{1}{n!} \Big[A_{ij}^{(n-2)} u_j^{(0)} + B_{ij}^{(n-2)} u_j^{(1)} + F_i^{(n-2)} \Phi^{(0)} + G_i^{(n-2)} \Phi^{(1)} \Big],$$
(3.9)

$$\Phi^{(n)} = \frac{1}{(n!)\epsilon_{zz}} \left[a_i^{(n-2)} u_i^{(0)} + b_i^{(n-2)} u_i^{(1)} + f^{(n-2)} \Phi^{(0)} + g^{(n-2)} \Phi^{(1)} \right],$$
(3.10)

where n = 0, 1, 2, ...

Eqs. (3.9) and (3.10) include recursively defined differential operators, given by (3.5) and the following extensions

$$\begin{split} A_{ik}^{(n)} &= B_{ij}^{(0)} A_{jk}^{(n-1)} + A_{ij}^{(0)} A_{jk}^{(n-2)} + \frac{1}{\epsilon_{zz}} [G_i^{(0)} a_k^{(n-1)} + F_i^{(0)} a_k^{(n-2)}], \\ B_{ik}^{(n)} &= B_{ij}^{(0)} B_{jk}^{(n-1)} + A_{ij}^{(0)} B_{jk}^{(n-2)} + \frac{1}{\epsilon_{zz}} [G_i^{(0)} b_k^{(n-1)} + F_i^{(0)} b_k^{(n-2)}], \\ F_i^{(n)} &= B_{ij}^{(0)} F_j^{(n-1)} + A_{ij}^{(0)} F_j^{(n-2)} + \frac{1}{\epsilon_{zz}} [G_i^{(0)} g^{(n-1)} + F_i^{(0)} g^{(n-2)}], \\ G_i^{(n)} &= B_{ij}^{(0)} G_j^{(n-1)} + A_{ij}^{(0)} A_{ij}^{(n-2)} + \frac{1}{\epsilon_{zz}} [g^{(0)} g^{(n-1)} + F_i^{(0)} g^{(n-2)}], \\ a_j^{(n)} &= b_i^{(0)} A_{ij}^{(n-1)} + a_i^{(0)} A_{ij}^{(n-2)} + \frac{1}{\epsilon_{zz}} [g^{(0)} a_j^{(n-1)} + f^{(0)} a_j^{(n-2)}], \\ b_j^{(n)} &= b_i^{(0)} B_{ij}^{(n-1)} + a_i^{(0)} B_{ij}^{(n-2)} + \frac{1}{\epsilon_{zz}} [g^{(0)} f_j^{(n-1)} + f^{(0)} f_j^{(n-2)}], \\ f^{(n)} &= b_i^{(0)} F_i^{(n-1)} + a_i^{(0)} F_i^{(n-2)} + \frac{1}{\epsilon_{zz}} [g^{(0)} g^{(n-1)} + f^{(0)} g^{(n-2)}], \\ g^{(n)} &= b_i^{(0)} G_i^{(n-1)} + a_i^{(0)} G_i^{(n-2)} + \frac{1}{\epsilon_{zz}} [g^{(0)} g^{(n-1)} + f^{(0)} g^{(n-2)}], \end{split}$$

for $n = 0, 1, 2, \ldots$, using the base terms

$$\begin{aligned} A_{ij}^{(-2)} &= \delta_{ij}, \qquad A_{ij}^{(-1)} &= 0, \qquad B_{ij}^{(-2)} &= 0, \qquad B_{ij}^{(-1)} &= \delta_{ij}, \\ F_i^{(-2)} &= 0, \qquad F_i^{(-1)} &= 0, \qquad G_i^{(-2)} &= 0, \qquad G_i^{(-1)} &= 0, \\ a_i^{(-2)} &= 0, \qquad a_i^{(-1)} &= 0, \qquad b_i^{(-2)} &= 0, \qquad b_i^{(-1)} &= 0, \\ f^{(-2)} &= \epsilon_{zz}, \qquad f^{(-1)} &= 0, \qquad g^{(-2)} &= 0, \qquad g^{(-1)} &= \epsilon_{zz}. \end{aligned}$$
(3.12)

Note that the recursion formulas (3.9) and (3.10) involve no approximations since they stem from the definition of the series expansions (3.1) and (3.2) and are as such exact. These may be obtained by using variational calculus as in [38]. Moreover, no truncations of the displacement terms have so far been performed, which is of crucial importance for the present method.

Insertion of the power series expansions (3.1) and (3.2) into the constitutive equations for the stresses (2.5) and for the electric displacements (2.6) gives

$$\Gamma_{I} = \sum_{n=0}^{\infty} z^{n} T_{I}^{(n)}, \qquad (3.13)$$

$$D_i = \sum_{n=0}^{\infty} z^n D_i^{(n)},$$
(3.14)

where

$$T_{I}^{(n)} = c_{IJ} \Big(\overline{\nabla}_{Jk} u_{k}^{(n)} + (n+1) L_{Jk} u_{k}^{(n+1)} \Big) \\ + e_{IJ} \Big(\overline{\nabla}_{J} \Phi^{(n)} + (n+1) l_{J} \Phi^{(n+1)} \Big),$$
(3.15)

$$D_{i}^{(n)} = e_{ij} \left(\overline{\nabla}_{jk} u_{k}^{(n)} + (n+1) L_{jk} u_{k}^{(n+1)} \right) - \epsilon_{ij} \left(\overline{\nabla}_{j} \Phi^{(n)} + (n+1) l_{j} \Phi^{(n+1)} \right).$$
(3.16)

By using the recursion formulas (3.9) and (3.10) these fields may finally be expressed in the lowest-order terms

$$\begin{split} T_{l}^{(n)} &= \frac{1}{n!} \left[\left[c_{lj} \left(\overline{\nabla}_{jk} A_{kl}^{(n-2)} + L_{jk} A_{kl}^{(n-1)} \right) + \frac{e_{lj}}{\epsilon_{zz}} \left(\overline{\nabla}_{j} a_{l}^{(n-2)} + l_{j} a_{l}^{(n-1)} \right) \right] u_{l}^{(0)} \\ &+ \left[c_{lj} \left(\overline{\nabla}_{jk} B_{kl}^{(n-2)} + L_{jk} B_{kl}^{(n-1)} \right) + \frac{e_{lj}}{\epsilon_{zz}} \left(\overline{\nabla}_{j} b_{l}^{(n-2)} + l_{j} b_{l}^{(n-1)} \right) \right] u_{l}^{(1)} \\ &+ \left[c_{lj} \left(\overline{\nabla}_{jk} F_{k}^{(n-2)} + L_{jk} F_{k}^{(n-1)} \right) + \frac{e_{lj}}{\epsilon_{zz}} \left(\overline{\nabla}_{j} f^{(n-2)} + l_{j} f^{(n-1)} \right) \right] \Phi^{(0)} \\ &+ \left[c_{lj} \left(\overline{\nabla}_{jk} G_{k}^{(n-2)} + L_{jk} G_{k}^{(n-1)} \right) + \frac{e_{lj}}{\epsilon_{zz}} \left(\overline{\nabla}_{j} g^{(n-2)} + l_{j} g^{(n-1)} \right) \right] \Phi^{(1)} \right], \end{split}$$

$$(3.17)$$

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$$\begin{split} D_{l}^{(n)} &= \frac{1}{n!} \bigg[\bigg[e_{ij} \Big(\overline{\nabla}_{jk} A_{kl}^{(n-2)} + L_{jk} A_{kl}^{(n-1)} \Big) - \frac{\epsilon_{ij}}{\epsilon_{zz}} \Big(\overline{\nabla}_{j} a_{l}^{(n-2)} + l_{j} a_{l}^{(n-1)} \Big) \bigg] u_{l}^{(0)} \\ &+ \bigg[e_{ij} \Big(\overline{\nabla}_{jk} B_{kl}^{(n-2)} + L_{jk} B_{kl}^{(n-1)} \Big) - \frac{\epsilon_{ij}}{\epsilon_{zz}} \Big(\overline{\nabla}_{j} b_{l}^{(n-2)} + l_{j} b_{l}^{(n-1)} \Big) \bigg] u_{l}^{(1)} \\ &+ \bigg[e_{ij} \Big(\overline{\nabla}_{jk} F_{k}^{(n-2)} + L_{jk} F_{k}^{(n-1)} \Big) - \frac{\epsilon_{ij}}{\epsilon_{zz}} \Big(\overline{\nabla}_{j} f^{(n-2)} + l_{j} f^{(n-1)} \Big) \bigg] \Phi^{(0)} \\ &+ \bigg[e_{ij} \Big(\overline{\nabla}_{jk} G_{k}^{(n-2)} + L_{jk} G_{k}^{(n-1)} \Big) - \frac{\epsilon_{ij}}{\epsilon_{zz}} \big(\overline{\nabla}_{j} g^{(n-2)} + l_{j} g^{(n-1)} \big) \bigg] \Phi^{(1)} \bigg]. \end{split}$$

$$(3.18)$$

Note that if all electric terms are disregarded in the relations presented in this section, the simplified expressions are in line with the work by Mauritsson et al. [35] on anisotropic elastic plates.

4. Plate equations

This section aims at deriving the dynamical plate equations through the surface boundary conditions at $z = \pm h$. From now on it is understood that the power series sums are to be truncated, where the truncation level could be chosen to any order. In order to write the expressions on a compact form using abbreviated subscripts, the prescribed fields for mechanical tractions and electric surface charges are here expressed in terms of mechanical stresses and electric displacements.

At each point on the surfaces $z = \pm h$, either the mechanical stress or the mechanical displacement is to be prescribed in each coordinate direction. These given fields are denoted by $\{T_i^+, u_i^+\}$ and $\{T_i^-, u_i^-\}$ at the upper and lower surfaces, respectively, where l = 5, 4, 3 and i = x, y, z. Hence, one of the fields for each of the six pairs $\{T_{xz}^\pm, u_x^\pm\}, \{T_{yz}^\pm, u_y^\pm\}, \{T_{zz}^\pm, u_z^\pm\}$ is to be given, resulting in three mechanical boundary conditions at each point on the upper and lower surfaces, respectively. Similarly, either the normal electric displacement component or the electric potential is to be prescribed at each point on the surfaces $z = \pm h$. These are expressed as $\{D_z^+, \Phi^+\}$ and $\{D_z^-, \Phi^-\}$ at the upper and lower surfaces, respectively, and result in one electric boundary conditions at each point on the surfaces. All in all four boundary conditions are thus to be stated at each surface point. Using (3.1), (3.2) and (3.13), (3.14) these are written

$$\sum_{n=0}^{N} h^{n} P^{(n)}(x,y,t) = P^{+}(x,y,t), \quad \sum_{n=0}^{N} (-h)^{n} P^{(n)}(x,y,t) = P^{-}(x,y,t), \quad (4.1)$$

$$\sum_{n=0}^{N+1} h^n p^{(n)}(x,y,t) = p^+(x,y,t), \quad \sum_{n=0}^{N+1} (-h)^n p^{(n)}(x,y,t) = p^-(x,y,t), \quad (4.2)$$

where $P = \{T_I, D_z\}$ and $p = \{u_i, \Phi\}$ for I = 5, 4, 3 and i = x, y, z. The difference in number of terms used for these equations is due to the differences in the differential orders. Since the mechanical stress terms $T_I^{(n)}$ in (3.15) and the electric displacement terms $D_i^{(n)}$ in (3.16) include spatial derivatives of one order higher than the mechanical displacements $u_i^{(n)}$ and the electric potentials $\Phi^{(n)}$, an extra term is to be included in the latter sums to obtain a consistent set of plate equations.

These surface boundary conditions obtained from combinations of (4.1) and (4.2) constitute the sought hyperbolic set of eight scalar equations of motion for a piezoelectric plate. By using the expansions (3.9), (3.10) and (3.17), (3.18) a hierarchy of piezoelectric plate equations are expressed in terms of the lowest order fields $\{u_x^{(0)}, u_y^{(0)}, u_z^{(0)}, \phi^{(0)}, u_x^{(1)}, u_y^{(1)}, u_z^{(1)}, \phi^{(1)}\}$. This system may be truncated to any order where the higher order sets can be used for obtaining high accuracy solutions, while the lower order sets may be used as engineering plate equations. Note that the resulting governing set of differential equations are in the general case fundamentally different for various parts of the plate, depending on the sort of mechanical boundary conditions (stresses or displacements) and the sort of electric boundary conditions (potentials or displacements) prescribed on a specific surface region. However, the surface boundary conditions are in all cases fulfilled exactly for the expansion order in question.

Each of the eight surface boundary conditions (4.1) and (4.2) involves spatial derivatives of order N+1 on $u_i^{(0)}$ and $\Phi^{(0)}$ (except when N = 0 for prescribed mechanical displacements and electric potentials) and spatial derivatives of order N on $u_i^{(1)}$ and $\Phi^{(1)}$. However, the resulting system of eight plate equations turns out to be of differential order 2(4N - 1) in the *xy* coordinates due to cancelation effects. This is readily seen by eliminating all but one of the fields, say w_0 , resulting in one equation of spatial order 2(4N-1). From this single equation, it is also seen that the number N should preferably be an odd integer. By choosing N odd, this equation includes space and time derivatives on w_0 up to differential order N + 1 which render asymptotically correct results [35]. For *N* even, the equation is only asymptotically correct up to differential order N. Furthermore, to account for the bending stiffness in the plate the smallest truncation order is N = 3, as this truncation yields fourth order derivatives with respect to the time and space coordinates on w_0 in the one single equation.

4.1. Comparisons to exact theory

The hierarchy of approximate plate equations may be compared analytically to the exact equations of motion by studying the corresponding frequency equations. Consider here the standard case of an infinite plate where the surface boundaries are stress free with zero potential. For the present approximate theory, separate frequency equations for antisymmetric and symmetric motions are obtained as polynomial expressions in terms of the wave numbers and the frequency. These results are to be compared to the exact antisymmetric and symmetric transcendental frequency equations, where the expressions in terms of wave numbers and frequency are expanded in Maclaurin series. Although comparisons could be performed to any order, only the case with truncation order N = 5 has been studied due to the complicated expressions. Hereby, each polynomial term up to and including order N = 5 in the approximate theories (antisymmetric and symmetric) is identical to the corresponding term in the Maclaurin series using exact theory. This illustrates that the present approach is asymptotically correct for these low order terms, and thus probably also to arbitrary order. Similar results are reported for longitudinal displacements in rods [37], beams [38] and for flexural elastic plates [40,44] when using a series expansion approach. Hence, the subsequent plate equations presented in Section 6 are thus asymptotically correct to order N = 3(antisymmetric) and N = 1 (symmetric), respectively.

5. Edge boundary conditions

This section aims at developing the 4N - 1 boundary conditions on each edge for a rectangular plate. This is performed in a systematic manner based on variational calculus.

Consider a rectangular plate where $-a \le x \le a$ and $-b \le y \le b$. As for the surfaces $z = \pm h$, either the mechanical stress or the mechanical displacement is to be prescribed in each coordinate direction over the entire thickness interval $-h \le z \le h$. At $x = \pm a$ these fields are denoted $\{T_i^{\pm a}, u_i^{\pm a}\}$ where I = 1, 6, 5 and i = x, y, z. Similarly, either the normal electric displacement component or the electric potential is to be prescribed, which at $x = \pm a$ are denoted $\{D_x^{\pm a}, \Phi^{\pm a}\}$. The edges $y = \pm b$ are treated analogously.

The edge boundary conditions may be obtained by adopting a generalized Hamilton's principle where mechanical stresses and mechanical displacements as well as the electric displacements and electric potentials are varied simultaneously and independently. Following the results from variational calculus presented in [37,38], the edge boundary conditions are obtained in a systematic manner. Since all boundary fields hereby are derived in the same manner, the procedure is here presented for a given normal stress at x = a. Assuming a prescribed stress $T_{xx}^{+a}(y, z, t)$, the variationally consistent boundary condition is expressed as [35]

$$\int_{-h}^{h} \left(T_{xx}^{+a}(y,z,t) - \sum_{n=0}^{M} z^n T_{xx}^{(n)}(a,y,t) \right) z^k dz = 0, \quad k = 0, 1, \dots, M,$$
(5.1)

adopting (3.13) to order *M*. It is thus straightforward to derive the M + 1 unknowns $T_{xx}^{(n)}$ from the system of M + 1 equations obtained from the performed integrals. The hereby obtained representation of the boundary stress function T_{xx} in power series is identical to the expansion of the given function T_{xx}^{+a} in terms of Legendre polynomials $P_n(z/h)$ of order *M*. Consequently

$$T_{xx}^{+a}(y,z,t) \approx \sum_{n=0}^{M} a_n(y,t) P_n(z/h) = \sum_{n=0}^{M} z^n T_{xx,n}^{+a}(y,t),$$
(5.2)

and hereby $T_{xx}^{(n)}(a, y, t) = T_{xx,n}^{+a}(y, t)$. Note that the standard case $T_{xx}^{+a} = 0$ results in $T_{xx}^{(n)} = 0$ for all n.

In line with the anisotropic elastic plate, the total number of 4N - 1 edge conditions on $x = \pm a$ are distributed as N boundary conditions (M = N - 1) on each of the three in plane pairs $\{T_{xx}^{(n)}, u_n\}, \{T_{xy}^{(n)}, v_n\}, \{D_x^{(n)}, \Phi^{(n)}\}$ while N - 1 boundary conditions (M = N - 2) on the out of plane pair $\{T_{xz}^{(n)}, w_n\}$. Corresponding results are obtained at $y = \pm b$. The orders, $k = 0, 1, \ldots$, are chosen in a manner to obtain the correct distribution of boundary conditions between the symmetric and antisymmetric parts.

In the case of connected homogeneous rectangular plates with different properties (geometrical, material, surface boundary conditions) the 2(4N - 1) coupling conditions at a common edge are deduced in a similar way as the edge boundary conditions see [35].

5.1. Calculation of fields

From the solution of the piezoelectric plate equations of motion (4.1) and (4.2) adopting the 4N - 1 boundary conditions on each edge, the plate fields are to be calculated by truncating the power series expansions (3.1), (3.2) and (3.13), (3.14). Since a lower truncation order is used for the edge boundary conditions than for the surface boundary conditions as described above, one has to make a choice regarding the adopted number of terms. Based on experience from [35,37], the same truncation procedure is used as for the governing plate Eqs. (4.1) and (4.2); N+2 terms for the mechanical displacements and the electric potential in (3.1) and (3.2), while N + 1 terms are adopted for the mechanical stresses and the electric displacements (3.13) and (3.14). Hereby the surface boundary conditions are exactly fulfilled while the edge boundary conditions are corrupted. However, close to or even at the edge boundaries, the discrepancies due to extra terms in the chosen approach are either negligible or small among the cases studied.

6. Orthotropic piezoelectric plate poled in the x direction

The preceding set of piezoelectric plate equations with pertinent boundary conditions are expressed in a general form that are applicable to all sorts of anisotropic configurations. By choosing the truncation order, plate equations of various differential order are obtained in a very systematic manner. However, in many applications the structure properties involve symmetry planes such as for orthotropic materials. Therefore consider the special case of a material of class 2mm poled in the x direction, i.e. the polarization axis lies in the same plane as the plate. Such a plate configuration has been studied by Tiersten [25] which calls for comparisons between theories. This turns the recursion formulas and the plate equations to a considerably more tractable form, where it is possible to study standard boundary problems separately for antisymmetric and symmetric motions. It is straightforward to truncate the equations to arbitrary order resulting in equations of high accuracy that can be used as benchmark solutions. By taking lower order truncation, engineering piezoelectric plate equations are obtained that may be compared to other approximate theories for orthotropic piezoelectric plates.

For the material configuration in question there are now 9 independent stiffness constants

$$[c_{ij}] = \begin{pmatrix} c_{11} & c_{12} & c_{13} & 0 & 0 & 0 \\ c_{12} & c_{22} & c_{23} & 0 & 0 & 0 \\ c_{13} & c_{23} & c_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & c_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & c_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & c_{66} \end{pmatrix},$$
(6.1)

3 independent dielectric constants

$$[\epsilon_{ij}] = \begin{pmatrix} \epsilon_{xx} & 0 & 0 \\ 0 & \epsilon_{yy} & 0 \\ 0 & 0 & \epsilon_{zz} \end{pmatrix}.$$
 (6.2)

and 5 independent piezoelectric coupling constants

$$[e_{ij}] = \begin{pmatrix} e_{x1} & e_{x2} & e_{x3} & 0 & 0 & 0\\ 0 & 0 & 0 & 0 & 0 & e_{y6}\\ 0 & 0 & 0 & 0 & e_{z5} & 0 \end{pmatrix}.$$
 (6.3)

The recursion formulas (3.3) and (3.4) are explicitly written

$$\begin{split} u_{n+2} &= \frac{1}{(n+1)(n+2)g_{55}} [\epsilon_{zz} \rho \partial_t^2 u_n - g_{11} \partial_x^2 u_n - g_{66} \partial_y^2 u_n \\ &- g_{126} \partial_x \partial_y v_n - (n+1) g_{135} \partial_x w_{n+1} - h_{15} \partial_x^2 \Phi_n - h_{65} \partial_y^2 \Phi_n], \\ v_{n+2} &= \frac{1}{(n+1)(n+2)c_{44}} [\rho \partial_t^2 v_n - c_{66} \partial_x^2 v_n - c_{22} \partial_y^2 v_n - c_{126} \partial_x \partial_y u_n \\ &- (n+1)c_{234} \partial_y w_{n+1} - e_{26} \partial_x \partial_y \Phi_n], \\ w_{n+2} &= \frac{1}{(n+1)(n+2)c_{33}} [\rho \partial_t^2 w_n - c_{55} \partial_x^2 w_n - c_{44} \partial_y^2 w_n \\ &- (n+1)c_{135} \partial_x u_{n+1} - (n+1)c_{234} \partial_y v_{n+1} - (n+1)e_{35} \partial_x \Phi_{n+1}], \\ \Phi_{n+2} &= \frac{1}{(n+1)(n+2)g_{55}} [e_{z5} \rho \partial_t^2 u_n - f_{11} \partial_x^2 u_n - f_{66} \partial_y^2 u_n \\ &- f_{126} \partial_x \partial_y v_n - (n+1)f_{13} \partial_x w_{n+1} - g_{51} \partial_x^2 \Phi_n - g_{56} \partial_y^2 \Phi_n], \end{split}$$

where n = 0, 1, 2, ..., and the fields are denoted $u_x^{(n)} = u_n$, $u_y^{(n)} = v_n, u_z^{(n)} = w_n$, and $\Phi^{(n)} = \Phi_n$ for convenience. In line with (3.9) and (3.10) these higher order fields may be expressed in the lowest order fields, not pursued here.

Several new constants are here introduced:

$$\begin{array}{ll} c_{126} = c_{12} + c_{66}, & c_{135} = c_{13} + c_{55}, & c_{234} = c_{23} + c_{44}, \\ e_{26} = e_{x2} + e_{y6}, & e_{35} = e_{x3} + e_{z5}, \\ f_{11} = c_{11}e_{z5} - c_{55}e_{x1}, & f_{13} = c_{13}e_{z5} - c_{55}e_{x3}, & f_{66} = c_{66}e_{z5} - c_{55}e_{y6}, \\ f_{126} = c_{126}e_{z5} - c_{55}e_{26}, & & \\ g_{11} = c_{11}\epsilon_{zz} + e_{z5}e_{x1}, & g_{66} = c_{66}\epsilon_{zz} + e_{z5}e_{y6}, \\ g_{51} = c_{55}\epsilon_{xx} + e_{z5}e_{x1}, & g_{56} = c_{55}\epsilon_{yy} + e_{z5}e_{y6}, \\ g_{126} = c_{126}\epsilon_{zz} + e_{z5}e_{26}, & g_{135} = c_{135}\epsilon_{zz} + e_{z5}e_{35}, \\ h_{15} = e_{x1}\epsilon_{zz} - e_{z5}\epsilon_{xx}, & h_{65} = e_{y6}\epsilon_{zz} - e_{z5}\epsilon_{yy}. \end{array}$$

The mechanical stress expansion functions $T_{ij}^{(n)}$ are obtained from (3.15) and these become

$$\begin{split} T_{xx}^{(n)} &= c_{11}\partial_{x}u_{n} + c_{12}\partial_{y}\upsilon_{n} + (n+1)c_{13}w_{n+1} + e_{x1}\partial_{x}\Phi_{n}, \\ T_{yy}^{(n)} &= c_{12}\partial_{x}u_{n} + c_{22}\partial_{y}\upsilon_{n} + (n+1)c_{23}w_{n+1} + e_{x2}\partial_{x}\Phi_{n}, \\ T_{zz}^{(n)} &= c_{13}\partial_{x}u_{n} + c_{23}\partial_{y}\upsilon_{n} + (n+1)c_{33}w_{n+1} + e_{x3}\partial_{x}\Phi_{n}, \\ T_{xz}^{(n)} &= c_{55}[(n+1)u_{n+1} + \partial_{x}w_{n}] + (n+1)e_{z5}\Phi_{n+1}, \\ T_{yz}^{(n)} &= c_{44}[(n+1)\upsilon_{n+1} + \partial_{y}w_{n}], \\ T_{xy}^{(n)} &= c_{66}(\partial_{y}u_{n} + \partial_{x}\upsilon_{n}) + e_{y6}\partial_{y}\Phi_{n}. \end{split}$$
(6.6)

Similarly, the electric displacements functions $D_i^{(n)}$ are obtained from (3.16)

$$D_{x}^{(n)} = e_{x1}\partial_{x}u_{n} + e_{x2}\partial_{y}v_{n} + (n+1)e_{x3}w_{n+1} - \epsilon_{xx}\partial_{x}\Phi_{n},$$

$$D_{y}^{(n)} = e_{y6}(\partial_{y}u_{n} + \partial_{x}v_{n}) - \epsilon_{yy}\partial_{y}\Phi_{n},$$

$$D_{z}^{(n)} = e_{z5}\partial_{x}w_{n} + (n+1)e_{z5}u_{n+1} - (n+1)\epsilon_{zz}\Phi_{n+1}.$$

(6.7)

The fields in (6.6) and (6.7) may be written in terms of the lowest order fields as in (3.17) and (3.18) using (6.4).

As noted before various combinations of the boundary conditions stated in (4.1) and (4.2) may be used. Consider for simplicity the case when either the mechanical stress or the mechanical displacement is prescribed on both the upper and lower surfaces for a specific direction. Similarly, assume that either the electric potential or the electric displacement is prescribed on both the upper and lower surfaces. By adding and subtracting the boundary conditions in (4.1) and (4.2), equations containing only even or odd expansions in *h* are obtained according to

$$\sum_{n=0}^{\lfloor N/2 \rfloor} h^{2n} P^{(2n)} = \frac{1}{2} (P^+ + P^-), \qquad \sum_{n=0}^{\lfloor (N-1)/2 \rfloor} h^{2n+1} P^{(2n+1)} = \frac{1}{2} (P^+ - P^-), \quad (6.8)$$
$$\sum_{n=0}^{\lfloor (N+1)/2 \rfloor} h^{2n} p^{(2n)} = \frac{1}{2} (p^+ + p^-), \qquad \sum_{n=0}^{\lfloor N/2 \rfloor} h^{2n+1} p^{(2n+1)} = \frac{1}{2} (p^+ - p^-), \quad (6.9)$$

where as before $P = \{T_{xz}, T_{yz}, T_{zz}, D_z\}$ and $p = \{u, v, w, \Phi\}$. Here $\lfloor n \rfloor$ denotes the floor of *n*, i.e. the nearest lower integer to *n*. These equations constitute decoupled sets corresponding to symmetrical and antisymmetrical motions. The *symmetric* part uses the first relation in (6.8) for T_{zz} and the second relation in (6.8) for $\{T_{xz}, T_{yz}, D_z\}$, and the first relation in (6.9) for $\{u, v, \Phi\}$ and the second relation in (6.9) for *w*. Hence, all possible combinations of these symmetric parts of the boundary conditions involve only the fields $\{u_0, v_0, w_1, \Phi_0\}$. Naturally the complementary combinations correspond to *antisymmetrical* sets involving $\{u_1, v_1, w_0, \Phi_1\}$.

For the sake of clarity, the plate equations of order N = 3 are presented explicitly below for antisymmetric motions while equations of order N = 1 are presented for symmetric motions. Hereby the antisymmetric motions correctly account for the flexural effects through the h^3 terms. For the mechanical surface boundary conditions, only the traditional case for prescribed stresses T_I^{\pm} are presented. The mechanical displacement boundary conditions are easily treated using (6.9) together with the recursion relations (6.4). As for the electrical surface boundary conditions, both types are given since potential boundary conditions Φ^{\pm} are usually stated for actuator problems while electric displacement boundary conditions D_z^{\pm} are usually stated for sensor problems. Concerning the edge boundary conditions, all combinations of mechanical and electrical fields are presented.

6.1. Antisymmetric motion N = 3

The plate equations for prescribed surface stresses are expressed in line with (6.8) using (6.6) and (6.4) as

$$c_{55}(u_{1} + \partial_{x}w_{0}) + e_{z5}\Phi_{1} + \frac{h^{2}}{2c_{33}}[c_{33}\rho\partial_{t}^{2}u_{1} - (c_{11}c_{33} - c_{13}c_{135})\partial_{x}^{2}u_{1} - c_{33}c_{66}\partial_{y}^{2}u_{1} - (c_{33}c_{126} - c_{13}c_{234})\partial_{x}\partial_{y}v_{1} - c_{13}\rho\partial_{t}^{2}\partial_{x}w_{0} + c_{13}c_{55}\partial_{x}^{3}w_{0} + c_{13}c_{44}\partial_{x}\partial_{y}^{2}w_{0} - (c_{33}e_{x1} - c_{13}e_{35})\partial_{x}^{2}\Phi_{1} - c_{33}e_{y6}\partial_{y}^{2}\Phi_{1}] = \frac{1}{2}(T_{xz}^{+} + T_{xz}^{-}), \qquad (6.10)$$

$$c_{44}(\nu_{1}+\partial_{y}w_{0}) + \frac{h^{2}}{2c_{33}} \Big[c_{33}\rho\partial_{t}^{2}\nu_{1} - (c_{22}c_{33} - c_{23}c_{234})\partial_{y}^{2}\nu_{1} - c_{33}c_{66}\partial_{x}^{2}\nu_{1} \\ - (c_{33}c_{126} - c_{23}c_{135})\partial_{x}\partial_{y}u_{1} - c_{23}\rho\partial_{t}^{2}\partial_{y}w_{0} + c_{23}c_{44}\partial_{y}^{3}w_{0} \\ + c_{23}c_{55}\partial_{x}^{2}\partial_{y}w_{0} - (c_{33}e_{26} - c_{23}e_{35})\partial_{x}\partial_{y}\Phi_{1} \Big] = \frac{1}{2}(T_{yz}^{+} + T_{yz}^{-}),$$

$$(6.11)$$

$$\begin{split} h[\rho\partial_{t}^{2}w_{0}-c_{55}(\partial_{x}u_{1}+\partial_{x}^{2}w_{0})-c_{44}(\partial_{y}v_{1}+\partial_{y}^{2}w_{0})-e_{z5}\partial_{x}\Phi_{1}] \\ &+\frac{h^{3}}{6c_{33}}\Big[\rho^{2}\partial_{t}^{4}w_{0}-c_{13}c_{55}\partial_{x}^{4}w_{0}-c_{23}c_{44}\partial_{y}^{4}w_{0}-(c_{13}c_{44}+c_{23}c_{55})\partial_{x}^{2}\partial_{y}^{2}w_{0} \\ &-(c_{55}-c_{13})\rho\partial_{t}^{2}\partial_{x}^{2}w_{0}-(c_{44}-c_{23})\rho\partial_{t}^{2}\partial_{y}^{2}w_{0}-(c_{33}+c_{135})\rho\partial_{t}^{2}\partial_{x}u_{1} \\ &+(c_{11}c_{33}-c_{13}c_{135})\partial_{x}^{3}u_{1}+[c_{33}(c_{126}+c_{66})-c_{23}c_{135}]\partial_{x}\partial_{y}^{2}u_{1} \\ &-(c_{33}+c_{234})\rho\partial_{t}^{2}\partial_{y}v_{1}+(c_{22}c_{33}-c_{23}c_{234})\partial_{y}^{3}v_{1}+[c_{33}(c_{126}+c_{66}) \\ &-c_{13}c_{234}]\partial_{x}^{2}\partial_{y}v_{1}-e_{35}\rho\partial_{x}\partial_{t}^{2}\Phi_{1}+(c_{33}e_{x1}-c_{13}e_{35})\partial_{x}^{3}\Phi_{1} \\ &+[c_{33}(e_{26}+e_{y6})-c_{23}e_{35}]\partial_{x}\partial_{y}^{2}\Phi_{1}\Big]=\frac{1}{2}(T_{zz}^{+}-T_{zz}^{-}). \end{split}$$

For prescribed electric potential or electric displacement, the corresponding equations read

$$h\Phi_{1} + \frac{h^{3}}{6c_{33}g_{55}}[c_{33}e_{25}\rho\partial_{t}^{2}u_{1} - (c_{33}f_{11} - c_{135}f_{13})\partial_{x}^{2}u_{1} - c_{33}f_{66}\partial_{y}^{2}u_{1} - (c_{33}f_{126} - c_{234}f_{13})\partial_{x}\partial_{y}v_{1} - f_{13}\rho\partial_{x}\partial_{t}^{2}w_{0} + c_{55}f_{13}\partial_{x}^{3}w_{0} + c_{44}f_{13}\partial_{x}\partial_{y}^{2}w_{0} - (c_{33}g_{51} - e_{35}f_{13})\partial_{x}^{2}\Phi_{1} - c_{33}g_{56}\partial_{y}^{2}\Phi_{1}] = \frac{1}{2}(\Phi^{+} - \Phi^{-}),$$
(6.13)

$$\begin{aligned} e_{z5}(u_{1}+\partial_{x}w_{0}) &- \epsilon_{zz} \varPhi_{1} + \frac{h^{2}}{2c_{33}} \Big[-(c_{33}e_{x1}-c_{135}e_{x3})\partial_{x}^{2}u_{1} - c_{33}e_{y6}\partial_{y}^{2}u_{1} \\ &- (c_{33}e_{z6}-c_{234}e_{x3})\partial_{x}\partial_{y}v_{1} - e_{x3}\rho\partial_{t}^{2}\partial_{x}w_{0} + c_{55}e_{x3}\partial_{x}^{3}w_{0} \\ &+ c_{44}e_{x3}\partial_{x}\partial_{y}^{2}w_{0} + (e_{x3}e_{35}+c_{33}\epsilon_{xx})\partial_{x}^{2}\varPhi_{1} + c_{33}\epsilon_{yy}\partial_{y}^{2}\varPhi_{1} \Big] \\ &= \frac{1}{2}(D_{z}^{+}+D_{z}^{-}). \end{aligned}$$

$$(6.14)$$

In order to solve for $\{u_1, v_1, w_0, \Phi_1\}$ from the set of four plate equations, (6.10)–(6.12) together with (6.13) or (6.14), the pertinent set of four boundary conditions at each edge is to be stated as described in Section 5. Consider the edge x = a, the boundary conditions for prescribed displacements and potential become

$$u_1^{+a}(y,t) = u_1, \quad v_1^{+a}(y,t) = v_1, \quad w_0^{+a}(y,t) = w_0, \quad \Phi_1^{+a}(y,t) = \Phi_1,$$
(6.15)

1.0

following the notation indicated in (5.2) using Legendre polynomial expansions of $\{u^{+a}(y,z,t), v^{+a}(y,z,t), w^{+a}(y,z,t), \Phi^{+a}(y,z,t)\}$. As for prescribed stresses and electric displacement, these boundary conditions become

$$T_{xx,1}^{+a}(y,t) = (c_{11} - \beta_{13}c_{135})\partial_x u_1 + (c_{12} - \beta_{13}c_{234})\partial_y v_1 + \beta_{13}\rho\partial_t^2 w_0 - \beta_{13}c_{55}\partial_x^2 w_0 - \beta_{13}c_{44}\partial_y^2 w_0 + (e_{x1} - \beta_{13}e_{35})\partial_x \Phi_1, T_{xy,1}^{+a}(y,t) = c_{66}(\partial_y u_1 + \partial_x v_1) + e_{y6}\partial_y \Phi_1, T_{xz,0}^{+a}(y,t) = c_{55}(u_1 + \partial_x w_0) + e_{25}\Phi_1, D_{xx,1}^{+a}(y,t) = (e_{x1} - \beta_{135}e_{x3})\partial_x u_1 + (e_{x2} - \beta_{234}e_{x3})\partial_y v_1 + \alpha\rho\partial_t^2 w_0 - \beta_{55}e_{x3}\partial_x^2 w_0 - \beta_{44}e_{x3}\partial_y^2 w_0 - (\epsilon_{xx} + \alpha e_{35})\partial_x \Phi_1,$$
(6.16)

where (6.6) and (6.7), the recursion relations (6.4) and the Legendre expansion (5.2) have been used. Naturally similar results are obtained for x = -a. For the edge y = b, the boundary conditions for prescribed displacements and potential become like in (6.15), while for prescribed stresses and electric displacement one has

$$\begin{aligned} T_{yy,1}^{+b}(x,t) &= (c_{12} - \beta_{23}c_{135})\partial_x u_1 + (c_{22} - \beta_{23}c_{234})\partial_y v_1 + \beta_{23}\rho \partial_t^2 w_0 \\ &- \beta_{23}c_{55}\partial_x^2 w_0 - \beta_{23}c_{44}\partial_y^2 w_0 + (e_{x2} - \beta_{23}e_{35})\partial_x \Phi_1, \\ T_{xy,1}^{+b}(x,t) &= c_{66}(\partial_y u_1 + \partial_x v_1) + e_{y6}\partial_y \Phi_1, \\ T_{yz,0}^{+b}(x,t) &= c_{44}(v_1 + \partial_y w_0), \\ D_{yy,1}^{+b}(x,t) &= e_{y6}(\partial_y u_1 + \partial_x v_1) - \epsilon_{yy}\partial_y \Phi_1. \end{aligned}$$
(6.17)

In (6.16) and (6.17) the following new constants are introduced $\alpha = e_{x3}/c_{33}, \quad \beta_{ij} = c_{ij}/c_{33}, \quad \beta_{ijk} = c_{ijk}/c_{33}.$ (6.18)

6.2. Symmetric motion N = 1

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The plate equations for prescribed surface stresses become from (6.8)

$$h[\rho\partial_t^2 u_0 - c_{11}\partial_x^2 u_0 - c_{66}\partial_y^2 u_0 - c_{126}\partial_x \partial_y v_0 - c_{13}\partial_x w_1 - e_{x1}\partial_x^2 \Phi_0 - e_{y6}\partial_y^2 \Phi_0] = \frac{1}{2}(T_{xz}^+ - T_{xz}^-),$$
(6.19)

$$h[\rho \partial_t^2 v_0 - c_{22} \partial_y^2 v_0 - c_{66} \partial_x^2 v_0 - c_{126} \partial_x \partial_y u_0 - c_{23} \partial_y w_1 - e_{26} \partial_x \partial_y \Phi_0] = \frac{1}{2} (T_{yz}^+ - T_{yz}^-),$$
(6.20)

$$c_{33}w_1 + c_{13}\partial_x u_0 + c_{23}\partial_y v_0 + e_{x3}\partial_x \Phi_0 = \frac{1}{2}(T_{zz}^+ + T_{zz}^-).$$
(6.21)

For prescribed electric potential or electric displacement, these equations are

$$\Phi_{0} + \frac{h^{2}}{2g_{55}} \Big[e_{z5}\rho \partial_{t}^{2} u_{0} - f_{11} \partial_{x}^{2} u_{0} - f_{66} \partial_{y}^{2} u_{0} - (c_{126}e_{z5} - c_{55}e_{26})\partial_{x}\partial_{y} v_{0} - g_{51} \partial_{x}^{2} \Phi_{0} - g_{56} \partial_{y}^{2} \Phi_{0} - f_{13} \partial_{x} w_{1} \Big] = \frac{1}{2} (\Phi^{+} + \Phi^{-}).$$
(6.22)

$$h\Big[-e_{x1}\partial_x^2 u_0 - e_{y6}\partial_y^2 u_0 - e_{z6}\partial_x \partial_y v_0 - e_{x3}\partial_x w_1 + \epsilon_{xx}\partial_x^2 \Phi_0 + \epsilon_{yy}\partial_y^2 \Phi_0\Big]$$

= $\frac{1}{2}(D_z^+ - D_z^-).$ (6.23)

There are three boundary conditions needed at each edge to solve for $\{u_0, v_0, w_1, \Phi_0\}$. Consider the edge x = a, the boundary conditions for prescribed displacements and potential become

$$u_0^{+a}(y,t) = u_0, \quad v_0^{+a}(y,t) = v_0, \quad \Phi_0^{+a}(y,t) = \Phi_0,$$
 (6.24)

while the stress and electric displacement boundary conditions are

$$\begin{aligned} T^{+a}_{xx_0}(y,t) &= c_{11}\partial_x u_0 + c_{12}\partial_y v_0 + c_{13}w_1 + e_{x1}\partial_x \Phi_0, \\ T^{+a}_{xy_0}(y,t) &= c_{66}(\partial_y u_0 + \partial_x v_0) + e_{y6}\partial_y \Phi_0, \\ D^{+a}_{xx_0}(y,t) &= e_{x1}\partial_x u_0 + e_{x2}\partial_y v_0 + e_{x3}w_1 - \epsilon_{xx}\partial_x \Phi_0. \end{aligned}$$
(6.25)

At the edge y = b, displacement and potential boundary conditions become like in (6.24), while for prescribed stresses and electric displacement one has

$$\begin{aligned} T^{+b}_{yy,0}(x,t) &= c_{12}\partial_{x}u_{0} + c_{22}\partial_{y}v_{0} + c_{23}w_{1} + e_{x2}\partial_{x}\Phi_{0}, \\ T^{+b}_{xy,0}(x,t) &= c_{66}(\partial_{y}u_{0} + \partial_{x}v_{0}) + e_{y6}\partial_{y}\Phi_{0}, \\ D^{+b}_{yy,0}(x,t) &= e_{y6}(\partial_{y}u_{0} + \partial_{x}v_{0}) - \epsilon_{yy}\partial_{y}\Phi_{0}. \end{aligned}$$
(6.26)

Note that all the results in this section are identical to the elastic orthotropic case in [35] if the electric coupling is disregarded.

These plate equations may be compared to other theories such as those of Kirchhoff and Mindlin type [25]. For the Kirchhoff theory only a few terms are similar to the present theory, while the Mindlin equations (involving two different correction factors) resemble the asymptotic plate equations in many respects. The low order derivative terms are similar in most cases, while the present theory involves higher order derivative terms that are not part of the Mindlin theory. However, similar higher order derivative terms are found in higher order theories as given analytically in [12,13,46], albeit the coefficients of these terms differ in most cases.

7. Numerical results

To investigate the accuracy of the present plate equations for both lower order engineering theories and the higher order sets, several numerical results are presented. The branches from the frequency equations (dispersion relations) for waves in an infinite plate poled in the *x* direction are illustrated, where distribution plots are presented for the displacements, stresses and electric potential for the lowest antisymmetric modes. Antisymmetric and symmetric eigenfrequencies are calculated for finite square simply supported plates poled in either the *x* or *z* directions. Moreover, eigenfrequencies for simply supported laminated piezoelectric plates are presented. The comparisons are made with exact three dimensional theory and other approximate theories presented in the literature.

7.1. Piezoelectric plate poled in the x direction

The piezoelectric plate is taken to be PZT-2 (Lead Zirconate Titanate) with x as the polarization axis. This is a 6mm material (a special case of 2mm) and elastically yz is a plane of isotropy. The material constants are (Auld [45]) $c_{11} = 113$ GPa, $c_{12} = c_{13} =$ 68.1 GPa, $c_{22} = c_{33} = 135$ GPa, $c_{23} = 67.0$ GPa, $c_{55} = c_{66} = 22.2$ GPa, $e_{x1} = 9.0 \text{ C/m}^2$, $e_{x2} = e_{x3} = -1.9 \text{ C/m}^2$, $e_{v6} = e_{z5} = 9.8 \text{ C/m}^2$, $\epsilon_{xx} =$ 260 ϵ_0 , and $\epsilon_{yy} = \epsilon_{zz} = 504\epsilon_0$. Here $\epsilon_0 = 8.854 \cdot 10^{-12} \text{ C/Vm}$ is the dielectric permittivity of free space. With yz as the plane of isotropy $c_{44} = \frac{1}{2}(c_{22} - c_{23})$. With this choice of material parameters the correction factors used in the Mindlin theory become $\kappa_4^2 = \pi^2/12$ and $\kappa_5^2 = 0.904$ [25]. The conditions are time harmonic with the time factor e^{-i ω t} and the dimensionless variable $\Omega = \omega h \sqrt{\rho/c_{33}}$ is used to measure the frequency. Consider free waves propagating in a direction 45° to the x axis. The piezoelectric layer is taken to be short-circuited ($\Delta V = 0$) and the electric potential in the middle of the plate is put to zero. The wave number in the direction of propagation is k.

In Fig. 2 the three first antisymmetric dispersion curves are plotted as dimensionless frequency Ω versus dimensionless wave number *kh*. The results from the asymptotic plate Eqs. (6.10)–(6.13) for N = 3 are compared with the exact theory, the Mindlin

theory and the Kirchhoff theory [25]. The first curve corresponds to the first bending mode with an almost parabolic behavior at low frequencies. All three plate theories approximate this curve quite well for low wave numbers, but for higher wave numbers the Kirchhoff theory is not that accurate as expected. The Mindlin theory approximates the first mode slightly better than the asymptotic plate theory. The other two modes, not modeled by the Kirchhoff theory, correspond to coupled P/SV (pressure/transverse shear) and the SH (horizontal shear) modes [45].

The second set of curves correspond to a pseudo-SH mode where the Mindlin and the present N = 3 theories give accurate approximations, although the Mindlin theory is better and predicts the cut-off frequency exactly. The third set of curves correspond to the second bending mode. Also this mode is approximated better by the Mindlin theory, at least for lower frequencies where the cut-off frequency is exactly predicted. From the dispersion curves it seems like the Mindlin theory generally gives more accurate results than the asymptotic plate theory of order N = 3. However, it should be noticed that the correction factors used in the Mindlin theory are chosen to obtain accurate approximations of the dispersion curves at low frequencies. This does not necessarily mean that the displacements and stresses are accurately predicted, which will be seen below. Furthermore, the asymptotic plate theory converges very quickly towards the exact solution if terms of higher order are taken into account. By choosing N = 5in the asymptotic plate theory the approximate dispersion curves are considerably more accurate, being very close to the exact curves (not illustrated here). For N = 7 the approximate dispersion curves become indistinguishable from the exact ones for the frequencies and wave numbers shown in Fig. 2. This feature also holds for the remaining plots Figs. 3–10.

Now consider a specific frequency $\Omega = 0.5$. According to the dispersion curves in Fig. 2 there is only one real solution to the wave number for this frequency ($kh \approx 1.44$ according to exact theory) and this solution corresponds to the first bending mode. The displacements, the electric potential and the stresses for the different theories are plotted as functions of the dimensionless thickness coordinate z/h in Figs. 3–6. The figures show that the electric potential, the displacements and the stresses are not very well approximated by the Kirchhoff theory, except for the vertical normal stress T_{zz} , which is quite well approximated. The Mindlin theory gives more accurate results for many of the fields, but actually poorer results for the vertical normal stress. The asymptotic plate theory renders much more accurate results for all fields, especially for the electric potential Φ , the vertical displacement w and the shear stresses T_{xz} . Here, the error in sign for Φ using Kirchhoff and Mindlin theories is of particular interest. However, similar behavior is also reported for the asymptotic theory in the case of piezoelectric layer on an elastic plate [42]. Note that both the Kirchhoff and the Mindlin theories assume constant vertical displacements, lying on top of the horizontal axis in Fig. 4(b). It should also be noticed that the boundary conditions for the vertical normal stress and the electric potential at the upper surface of the plate are not fulfilled by the Mindlin or the Kirchhoff theory, but are exactly fulfilled by the asymptotic plate theory. As the results for $\{v, T_{yy}, T_{yz}\}$ resemble $\{u, T_{xx}, T_{xz}\}$ the former fields are not illustrated.

Note that for the Mindlin and the Kirchhoff theory the stresses T_{xx} , T_{yy} and T_{xy} are determined from the constitutive Eq. (2.5) (the direct method), while the stresses in the vertical direction T_{zz} , T_{yz} and T_{xz} are obtained using the indirect method. This is accomplished by inserting the stresses and displacements from the direct method into the equations of motion (2.1) and integrating with respect to *z*. The integration constants are determined by requiring that the shear stresses, T_{yz} and T_{xz} , are zero at the plate surfaces ($z = \pm h$) and that the normal stress, T_{zz} , is zero in the middle of



Fig. 2. Dispersion curves: -Exact, ---N = 3, \cdots Mindlin, -- Kirchhoff.



Fig. 3. Electric potential for the first bending mode, $\Omega = 0.5$: -Exact, ---N = 3, \cdots Mindlin, -- Kirchhoff.

the plate (z = 0) due to antisymmetry. This indirect way of calculating the stresses, also used by Reddy [47], gives more accurate results compared to calculating all stress components from the constitutive equations, which only yields constant or linear variations in the thickness coordinate. If the indirect method is used for the stresses using the asymptotic plate theory, the same results are obtained after truncation as with the direct method. Either the real or the imaginary part of each quantity is shown in the figures as only one of them differs from zero. Moreover, due to symmetry/ antisymmetry, only the upper half of the plate $(0 \le z \le h)$ is shown. The fields are normalized so that the vertical displacement in the middle of the plate equals unity, w(z = 0) = 1. The electric potential is measured with the dimensionless variable $\phi = \epsilon_{zz}/(he_{z5})\Phi$ and the stresses are normalized with the stiffness constant c_{33} .

Consider also the frequency $\Omega = 1$. According to the dispersion curves, there are three real solutions to the wave number for this frequency. In addition to the first bending mode solution there is also one solution for the pseudo-SH mode ($kh \approx 1.50$) and one for the second bending mode ($kh \approx 0.79$). Some of the fields for the latter two of these modes are plotted in Figs. 7-10, showing the electric potential, the dominating horizontal displacement, the vertical normal stress and the dominating shear stress for the two cases. The fields are normalized so that the imaginary part of the dominating horizontal displacement (u and v, respectively) equals unity at the upper surface of the plate. Also here, the asymptotic plate theory gives more accurate results than the Mindlin theory, which is not accurate at all for the electric potential or any of the stresses shown in the figures. However, it should be mentioned that the way of normalizing may affect the impression of how well different fields are approximated by different theories. In addition, if the truncation order is increased a few levels, the asymptotic



direction.



Fig. 4. Horizontal displacements for the first bending mode, $\Omega = 0.5$: - Exact, --- N = 3, ··· Mindlin, --- Kirchhoff.



Fig. 5. Stresses for the first bending mode, $\Omega = 0.5$: - Exact, --- $N = 3, \cdots$ Mindlin, --- Kirchhoff.



Fig. 6. Stresses for the first bending mode, $\Omega = 0.5$: - Exact, --- N = 3, ··· Mindlin, --- Kirchhoff.



Fig. 7. Electric potential and displacement for the pseudo-SH mode, $\Omega = 1$: - Exact, --- $N = 3, \cdots$ Mindlin.



Fig. 8. Stresses for the pseudo-SH mode, $\Omega = 1$: - Exact, --- N = 3, · · · Mindlin.



Fig. 9. Electric potential and displacement for the second bending mode, $\Omega = 1$: - Exact, --- N = 3, · · · Mindlin.



Fig. 10. Stresses for the second bending mode, $\Omega = 1$: - Exact, --- N = 3, · · · Mindlin.

plate theory curves become indistinguishable from the exact ones for all the displayed plots.

In order to further illustrate the present theory, the eigenfrequencies for simply supported quadratic plates are calculated. For exact and series expansion theories, this corresponds to mixed boundary conditions at the edges where normal stresses as well as vertical and tangential displacements are zero. Due to the polarization in the *x* direction, the electric field is such that the normal electric field displacement is zero at $x = \pm a$ while the potential is zero at $y = \pm b$. The Tables 1–4 illustrate the three lowest eigenfrequencies Ω_i for short-circuited antisymmetric and symmetric motions, respectively. Here three different plate thicknesses a/h are studied for two different mode cases, where *m* and *n* refer to the mode numbers in the *x* and *y* directions, respectively.

Consider first the antisymmetric cases presented in Tables 1 and 2. It is clear that the series expansion results converge to the

exact results as the power series orders are increased. The convergence is quite rapid considering the thick plates studied. The accuracies are higher for lower frequencies as expected, while the plate thickness itself does not much influence the rate of convergence. The Kirchhoff theory models the first eigenfrequency, where the accuracy highly depends on the plate thickness. The Mindlin theory is astonishingly accurate, at least for the lower frequencies. As noted before for the dispersion curves, the Mindlin theory involves the shear correction factors that are chosen so as to give good correlations in the low frequency interval. When compared to the N = 3 theory, the Mindlin results are generally superior for the case m = 1 and n = 1; especially at lower frequencies. For m = 1 and n = 2 the accuracy pattern is less apparent among the two theories.

The symmetric cases are given in Tables 3 and 4. As for the antisymmetric cases, the series expansion results converge quite

Table 1

The antisymmetric eigenfrequencies for zero surface potential when m = 1 and n = 1 using exact, Kirchhoff (K), Mindlin (M) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	Ω	Exact	K	М	<i>N</i> = 3	N = 5	<i>N</i> = 7	N = 9
10	Ω_1	0.02177	0.02254	0.02176	0.02168	0.02177	0.02177	0.02177
	Ω_2	0.6837	-	0.6833	0.6437	0.6862	0.6836	0.6837
	Ω_3	0.8051	-	0.8063	0.7270	0.8163	0.8046	0.8051
4	Ω_1	0.1179	0.1412	0.1176	0.1157	0.1178	0.1179	0.1179
	Ω_2	0.7436	-	0.7406	0.7038	0.7465	0.7435	0.7436
	Ω_3	0.8962	-	0.9045	0.8213	0.9092	0.8956	0.8962
1	Ω_1	0.8609	2.3325	0.8501	0.7904	0.8530	0.8603	0.8609
	Ω_2	1.262	-	1.253	1.253	1.266	1.262	1.262
	Ω_3	1.786	-	2.004	1.610	1.854	1.774	1.787

Table 2

The antisymmetric eigenfrequencies for zero surface potential when m = 1 and n = 2 using exact, Kirchhoff (K), Mindlin (M) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	Ω	Exact	K	Μ	<i>N</i> = 3	N = 5	N = 7	<i>N</i> = 9
10	Ω_1	0.05470	0.05929	0.05454	0.05417	0.05469	0.05470	0.05470
	Ω_2	0.6925	-	0.6905	0.6545	0.6949	0.6924	0.6925
	Ω_3	0.8448	-	0.8492	0.7663	0.8572	0.8443	0.8448
4	Ω_1	0.2617	0.3716	0.2589	0.2528	0.2615	0.2617	0.2617
	Ω_2	0.7912	-	0.7813	0.7661	0.7929	0.7912	0.7912
	Ω_3	1.082	-	1.109	0.9949	1.105	1.080	1.082
1	Ω_1	1.533	6.115	1.487	1.264	1.486	1.526	1.532
	Ω_2	1.805	-	1.830	1.902	1.840	1.813	1.806
	Ω_3	2.328	-	3.143	2.067	2.374	2.305	2.330

Table 3

The symmetric eigenfrequencies for zero surface potential when m = 1 and n = 1 using exact, Mindlin (M) and the series expansion theories of orders N = 1, 3, 5, 7, 9.

a/h	Ω	Exact	М	N = 1	<i>N</i> = 3	N = 5	N = 7	<i>N</i> = 9
10	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	0.1001 0.1761 1.524	0.1001 0.1765 -	0.1001 0.1766 -	0.1001 0.1761 1.218	0.1001 0.1761 1.607	0.1001 0.1761 1.506	0.1001 0.1761 1.525
4	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	0.2512 0.4374 1.463	0.2512 0.4334 -	0.2518 0.4448 -	0.2512 0.4372 1.182	0.2512 0.4374 1.725	0.2512 0.4374 1.446	0.2512 0.4374 1.463
1	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	1.035 1.271 1.903	1.029 1.838 -	1.049 1.904 -	1.038 1.222 1.681	1.036 1.280 1.959	1.035 1.271 1.892	1.035 1.271 1.903

Table 4 The symmetric eigenfrequencies for zero surface potential when m = 1 and n = 2 using exact, Mindlin (M) and the series expansion theories of orders N = 1, 3, 5, 7, 9.

-				-				
a/h	Ω	Exact	М	N = 1	<i>N</i> = 3	N = 5	N = 7	N = 9
10	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	0.1523 0.2923 1.486	0.1523 0.2938 -	0.1523 0.2940 -	0.1523 0.2922 1.194	0.1523 0.2923 1.642	0.1523 0.2923 1.468	0.1523 0.2923 1.488
4	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	0.3899 0.7095 1.431	0.3893 0.7379 -	0.3956 0.7405 -	0.3902 0.7053 1.175	0.3899 0.7097 1.572	0.3899 0.7095 1.416	0.3899 0.7095 1.431
1	$egin{array}{c} \Omega_1 \ \Omega_2 \ \Omega_3 \end{array}$	1.708 1.715 2.390	1.667 3.002 -	1.796 3.083 -	1.748 1.778 2.277	1.719 1.740 2.305	1.710 1.719 2.386	1.708 1.715 2.390

rapidly to the exact results as the power series orders are increased. The N = 1 results are of the same accuracy order as Mindlin's, albeit the latter are marginally better in most cases.

7.2. Piezoelectric plate poled in the z direction

Consider next a PZT-4 plate poled in the *z* direction. The material constants are [27] $E_1 = E_2 = 81.3$ GPa, $E_3 = 64.5$ GPa, $G_{12} = 30.6$ GPa, $G_{13} = G_{23} = 25.6$ GPa, $v_{12} = 0.329$, $v_{13} = v_{23} = 0.432$, $e_{x5} = e_{y4} = 12.72$ C/m², $e_{z1} = e_{z2} = -5.2$ C/m², $e_{z3} = 15.08$ C/m², $\epsilon_{xx} = \epsilon_{yy} = 1475\epsilon_0$, and $\epsilon_{zz} = 1300\epsilon_0$ where $\epsilon_0 = 8.85 \cdot 10^{-12}$ C/Vm. Here $c_{66} = (c_{12} - c_{22})/2$ is only approximately fulfilled using these material constants. Adopting the general anisotropic plate equations derived in Section 4, sets of antisymmetric and symmetric plate equations are obtained in a straightforward manner. These plate equations are different to the ones poled in the *x* direction presented in Section 6, although both poled directions involve terms of the same differential orders. Note that the plate equations poled in the *z* direction are also asymptotically correct to the studied order N = 5.

Consider simply supported quadratic plates, where the elastic boundary conditions are as in Section 7.1 while the potential is zero at all edges. Tables 5-8 illustrate the three lowest eigenfrequencies $\tilde{\omega}_i = \omega_i / 100$ when m = n = 1 for both antisymmetric and symmetric motions, using three different plate thicknesses a/h. Adopting this frequency normalization, the plate geometry is to be stated which here is set to 2h = 0.01. Comparisons are made to Benjeddou and Deü [48] and Robaldo et al. [20] for some cases studied therein. The theory in [48] is based on Mindlin assumptions without shear correction factors and thus should preferably be compared to the present theory of order N = 3 in the antisymmetric case and N = 1 in the symmetric case. The plate theory in [20] considers a series expansion theory of exponential order 4 solved using FEM, and thus may well be compared to the present theory for N = 4. The presented lists of eigenfrequencies involve quite a few significant digits considering the approximate nature of the given material and physical constants. The reason for this K. Mauritsson, P.D. Folkow/Computers and Structures 153 (2015) 112-125

Table 5

The antisymmetric eigenfrequencies for zero surface potential using exact, Benjeddou and Deü (BD), Robaldo et al. (RCB) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	õ	Exact	BD	RCB	N = 3	N = 5	N = 7	<i>N</i> = 9
50	$\tilde{\omega}_1$	746.752	746.837	746.834	746.581	746.730	746.752	746.752
	$\tilde{\omega}_2$	502,895			452,815	509,821	502,610	502,903
	$\tilde{\omega}_3$	503,469			453,359	510,418	503,182	503,477
10	$\tilde{\omega}_1$	18013.4	18071.1		17919.6	18012.6	18013.4	18013.4
	$\tilde{\omega}_2$	508,625			459,171	515,474	508,343	508,633
	$\tilde{\omega}_3$	522,320			472,052	529,764	522,000	522,329
4	$\tilde{\omega}_1$	96929.9	98246.6	97321.4	94718.4	96892.0	96929.6	96929.9
	$\tilde{\omega}_2$	538,885			492,479	545,354	538,619	538,892
	$\tilde{\omega}_3$	609,186			556,144	619,754	608,653	609,205

Table 6

The symmetric eigenfrequencies for zero surface potential using exact, Benjeddou and Deü (BD), Robaldo et al. (RCB) and the series expansion theories of orders N = 1, 3, 5, 7, 9.

a/h	ω	Exact	BD	RCB	N = 1	N = 3	N = 5	N = 7	<i>N</i> = 9
50	$\tilde{\omega}_1$	15540.4	15540.4	15542.0	15540.4	15540.4	15540.4	15540.4	15540.4
	ω_2	26828.0	26831.7	26828.2	26832.6	26828.0	26828.0	26828.0	26828.0
	$\tilde{\omega}_3$	1,004,344			-	783,502	1,094,561	984,390	1,006,587
10	$\tilde{\omega}_1$	77702.1	77702.1		77702.1	77702.1	77702.1	77702.1	77702.1
	$\tilde{\omega}_2$	133,695	134,158		134,273	133,691	133,695	133,695	133,695
	$\tilde{\omega}_3$	988,021			_	776,834	1,114,956	970,449	989,934
4	$\tilde{\omega}_1$	194,255	194,255	194,264	194,255	194,255	194,255	194,255	194,255
	$\tilde{\omega}_2$	327,663	335,396	327,670	337,105	327,245	327,674	327,663	327,663
	$\tilde{\omega}_3$	958,922			_	766,465	1,018,717	944,340	960,404

Table 7

The antisymmetric eigenfrequencies for zero normal surface electric displacement using exact, Benjeddou and Deü (BD) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	õ	Exact	BD	<i>N</i> = 3	N = 5	<i>N</i> = 7	N = 9	
50	$\tilde{\omega}_1$	746.873	746.957	746.702	746.873	746.873	746.873	
	$\tilde{\omega}_2$	502,895		452,815	509,821	502,610	502,903	
	$\tilde{\omega}_3$	586,240		512,371	603,332	585,385	586,227	
10	$\tilde{\omega}_1$	18077.8	18136.1	17982.9	18077.5	18077.8	18077.8	
	$\tilde{\omega}_2$	508,625		459,171	515,474	508,343	508,633	
	$\tilde{\omega}_3$	604,752		530,530	623,173	603,805	604,736	
4	$\tilde{\omega}_1$	98231.7	99634.9	95938.1	98195.5	98231.4	98231.7	
	$\tilde{\omega}_2$	538,885		492,479	545,354	538,619	538,892	
	$\tilde{\omega}_3$	690,767		612,873	717,862	689,262	690,735	

Table 8

The symmetric eigenfrequencies for zero normal surface electric displacement using exact, Benjeddou and Deü (BD) and the series expansion theories of orders N = 1, 3, 5, 7, 9.

a/h	õ	Exact	BD	N = 1	<i>N</i> = 3	N = 5	<i>N</i> = 7	<i>N</i> = 9
50	$egin{array}{c} ilde{\omega}_1 \ ilde{\omega}_2 \ ilde{\omega}_3 \end{array}$	15540.4 29153.3 1,004,612	15540.4 29155.6	15540.4 29157.8 -	15540.4 29153.3 783,507	15540.4 29153.3 1,035,593	15540.4 29153.3 984,552	15540.4 29153.3 1,006,872
10	$egin{array}{c} ilde{\omega}_1 \ ilde{\omega}_2 \ ilde{\omega}_3 \end{array}$	77702.1 145,221 990,953	77702.1 145,610	77702.1 145,789 -	77702.1 145,215 776,867	77702.1 145,221 1,030,751	77702.1 145,221 972,401	77702.1 145,221 993,008
4	$egin{array}{c} \tilde{\omega}_1 \ \tilde{\omega}_2 \ \tilde{\omega}_3 \end{array}$	194,255 355,110 960,103	194,255 361,926	194,255 364,473 -	194,255 354,338 766,832	194,255 355,133 1,013,234	194,255 355,109 945,063	194,255 355,110 961,646

is to make comparisons more transparent since such lists are presented in [20,48].

Tables 5 and 6 illustrate the three lowest eigenfrequencies for short-circuited antisymmetric and symmetric motions, respectively. The accuracies using the present series method are of the same order as for the plates poled in the x direction presented in Section 7.1. The series results converge to the exact results in all

cases studied, where a few more terms are needed for the higher frequencies (up to N = 13). As for the other theories, the Benjeddou and Deü theory is slightly more accurate for the eigenfrequencies than the present theory for N = 3 (antisymmetric) and N = 1 (symmetric), respectively. By increasing the series order one step to N = 5 (antisymmetric) and N = 3 (symmetric), the present results are improved considerably. As for the results due to

Table 9

Eigenfrequencies for laminated plate with zero normal surface potential using exact, Benjeddou and Deü (BD) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	õ	Exact	BD	<i>N</i> = 3	N = 5	N = 7	N = 9	
50	$\tilde{\omega}_1$	633.417	633.666	633.192	633.393	633.415	633.417	
	$\tilde{\omega}_2$	16431.1	16432.5	16430.7	16431.1	16431.1	16431.1	
	$\tilde{\omega}_3$	28535.2	28537.6	28534.0	28535.2	28535.2	28535.2	
	$\tilde{\omega}_4$	268,118	292,032	238,030	271,728	267,900	268,138	
4	$\tilde{\omega}_1$	72174.4	73969.5	70474.3	72122.2	72171.9	72174.3	
	$\tilde{\omega}_2$	194,760	197,165	193,744	194,770	194,760	194,760	
	$\tilde{\omega}_3$	306,209	329,399	302,768	306,159	306,205	306,209	
	$ ilde{\omega}_4$	337,107	345,027	301,741	340,906	336,821	337,134	

Table 10

Eigenfrequencies for laminated plate with zero normal surface electric displacement using exact, Benjeddou and Deü (BD) and the series expansion theories of orders N = 3, 5, 7, 9.

a/h	õ	Exact	BD	<i>N</i> = 3	N = 5	N = 7	<i>N</i> = 9	
50	$\tilde{\omega}_1$	633.487	633.735	633.280	633.469	633.486	633.487	
	$\tilde{\omega}_2$	16440.9	16442.2	16440.6	16440.9	16440.9	16440.9	
	$\tilde{\omega}_3$	28555.3	28557.7	28554.2	28555.3	28555.3	28555.3	
	$\tilde{\omega}_4$	271,222	295,866	240,684	274,472	271,007	271,245	
4	$\tilde{\omega}_1$	72191.5	74,006	70483.8	72141.7	72187.8	72191.3	
	$\tilde{\omega}_2$	194,881	197,280	194,072	194,889	194,881	194,881	
	$\tilde{\omega}_3$	306,539	329,862	302,839	306,501	306,534	306,539	
	$\tilde{\omega}_4$	337,196	345,226	301,923	340,508	336,930	337,223	

Robaldo et al., these show accuracies similar to N = 3 for thin plates and closer to N = 5 for thick plates. Eigenfrequencies for open-circuited plates are presented in Tables 7 and 8. The results show a similar pattern as for the short-circuited case.

It should be noted that the calculated exact eigenfrequencies presented in Tables 5–8 are identical to the results due to Heyliger and Saravanos [27].

7.3. Piezoelectric laminate

In order to further illustrate the present expansion method, eigenfrequencies for laminated piezoelectric plates are calculated. There are much work on various laminated plate structures with piezoelectric layers bonded to the plate faces [19-21,24,27,32, 33,46,48]. Among these, Heyliger and Saravanos [27] present exact 3D results for three and five layered simply supported plates. Here three-layer piezoelectric plates are considered, where comparisons are made to the exact theory [27] and the approximate Mindlin theory due to Benjeddou and Deü [48]. Consider a plate with total thickness 2h that consists of a PZT-4 core of thickness h (material constants as in Section 7.2) surrounded symmetrically by orthotropic PVDF layers with material constants [27] $E_1 = 237,0$ GPa, $E_2 = 23.2$ GPa, $E_3 = 10.5$ GPa, $G_{12} = 6.43$ GPa, $G_{13} = 4.4$ GPa, $G_{23} = 6.43$ GPa, $G_{13} = 4.4$ GPa, $G_{23} = 6.43$ GPa, $G_{13} = 6.43$ 2.15 GPa, $v_{12} = 0.154$, $v_{13} = 0.178$, $v_{23} = 0.177$, $e_{x5} = e_{y4} = 0.177$ $-0.01 \text{ C}/m^2$, $e_{z1} = -0.13 \text{ C}/m^2$, $e_{z2} = -0.14 \text{ C}/m^2$, $e_{z3} =$ $-0.28 \text{ C}/m^2$, $\epsilon_{xx} = 12.5\epsilon_0, \epsilon_{yy} = 11.98\epsilon_0$, and $\epsilon_{zz} = 11.98\epsilon_0$ where $\epsilon_0 = 8.85 \cdot 10^{-12} \text{ C/Vm.}$

The derivation process for the present series expansion techniques follows from Mauritsson et al. [42] applied to plates with piezoelectric core material. By splitting the solution process in symmetric and antisymmetric parts, eigenfrequencies are obtained for different truncation orders. The procedure is more involved than for a single plate layer, as separate series expansion systems are introduced for the layers. The coupling conditions at the layer interfaces are modeled at each point, and the free surface conditions are fulfilled at $z = \pm h$ just as for the single plate layer.

The four lowest eigenfrequencies $\tilde{\omega}_i = \omega_i/100$ when m = n = 1 and 2h = 0.01 are presented in Tables 9 and 10 for short-circuited

and open-circuited laminated plates, respectively. Both cases show similar results regarding eigenfrequencies and accuracies for the different approximative theories. In each case, the first and fourth eigenfrequencies correspond to antisymmetric motion while the middle frequencies are for symmetric motion. The series expansion approximations show similar behavior as for the single layer cases in Tables 5–8, albeit the rates of convergence are slightly lower in the laminated cases. This effect is most notable for the first symmetric mode $\tilde{\omega}_2$ for the thicker plate. Hence, the results indicate that the present method converges to the exact results for all the cases studied, although higher order truncations N > 9 have not been studied. Similar to the single plate layer, the results due to the Benjeddou and Deü theory are slightly more accurate for antisymmetric motion when compared to the present theory of order N = 3. The opposite holds for symmetric motions.

8. Concluding remarks

Plate equations and corresponding edge boundary conditions for a fully anisotropic piezoelectric rectangular plate are derived to arbitrary order using a systematic power series expansion approach. The plate equations are given on a compact form using recursively defined matrix differential operators, where the set of equations are variationally consistent and asymptotically correct to all studied orders. These equations are simplified for an orthotropic 2mm material poled in an in-plane direction, where the plate equations and edge boundary conditions are given explicitly. Numerical comparisons for dispersion curves and the cross sectional fields (potential, displacements, stresses) are made between the asymptotic plate theory, the Kirchhoff theory, the Mindlin theory and exact theory. Eigenfrequencies are presented for simply supported plates poled either in the plate plane or normal to the plate plane. Both single and laminated plates are studied for different plate thicknesses using various plate theories. The results illustrate the accuracies using different series truncations for the various eigenmodes where the rate of convergence is high in most cases, including the more complicated laminated plate structures.

The derived system of plate equations may either be used to low truncation order as engineering plate equations, or to higher order acting as three dimensional benchmark results. A possible application for the former case is to develop piezoelectric plate element for finite element codes. Hereby one benefits from the accurate results using one of the present lower order theories, and at the same time the number of elements can be heavily reduced compared to using three dimensional elements. As for the latter case using higher order truncations, this calls for an alternative way of obtaining solutions to the three dimensional equations [49]. This is especially of interest for more complicated anisotropic configurations.

Future work is to develop higher order theories for more complicated material configurations, such as involving functionally graded, porous or micro materials. For such configurations, several different plate theories have appeared, and the present systematic approach would render equations that are variationally consistent and directly based on the three dimensional theories. Related work based on power series expansion and recursion relations have previously been carried out on isotropic materials such as porous [50] and functionally graded [49] plates, and is currently directed towards micro plates. Another area of interest is to further study laminates involving multiple elastic plates embedded in piezoelectric layers [33]. This may be accomplished by adopting the present piezoelectric plate equations and the corresponding equations for anisotropic plates [35], using generalizations of the methodology presented in [42,43].

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