

$$1.4586r^3 - 3.6124r^2 + 0.7531r = 0$$

$$r = 0.231$$

$$\lambda_1 \left\{ r - \frac{4}{3} \times \frac{11}{191}(1-r) + \frac{r(1-r)}{0.13384} \times 0.0298317 \right\} + \lambda_r \left\{ \frac{4}{3} \times \frac{11}{191} + \frac{r}{0.13384} \times 0.236478 \right\} = 0$$

$$\lambda_1 \left\{ \frac{4}{3} \times \frac{11}{191}(1-r) + \frac{r(1-r)}{0.13384} \times 0.236478 \right\} + \lambda_r \left\{ -\frac{3}{4} \times \frac{11}{191} + \frac{r}{0.13384} \times 0.372014 \right\} = 0$$

$$\lambda_1 \{ r - 0.0767(1-r) + r(1-r) \times 0.2228 \} + \lambda_r \{ 0.0767 + 1.7658r \} = 0$$

$$\lambda_1 \{ 0.0767(1-r) + 1.7668r(1-r) + \lambda_r \{ -0.0767 + 2.778r \} = 0$$

Similarity

$$\therefore r = 0.391$$

then,

$$k_1 = 2.16 \times 10^6 \quad I_1 = 0.1338 \quad r = 0.231$$

$$P_1 = \sqrt{\frac{0.231 \times 2.16 \times 10^6}{0.1338}} = 1931.1$$

$$f_1 = \frac{1931.1}{2 \times 3.1416} = 307.34 \text{ cycle/sec}$$

$$k_2 = 4.51 \times 10^6 \quad I_2 = 0.1338 \quad r_2 = 0.391$$

$$P_2 = \sqrt{\frac{0.391 \times 4.51 \times 10^6}{0.1338}} = 3630.3$$

$$f_2 = \frac{3630.3}{2 \times 3.1416} = 577.78 \text{ cycle/sec}$$

By comparing  $f_1$  and  $f_2$  we gain the following conclusion,

### (3) Conclusion

In this discussion the free torsional vibration of shaft was considered and the frequencies of the natural vibrations were determined. When we changed the position of cams at the nearest position of the bearing, the torsional vibrations shall have the least value of the vibration and then the least impact on the shaft.

We believed that this changed positions of cams shall have the best conditions for the vibration by calculating the above two example.

## On the physical properties under the chemical treatment of leathers.

K. OKUDA, I. TSUJIMOTO.

### (An Abstract)

In this discussion, our objects are satisfied by researching the chemical and physical properties of leathers on their stress-strain curves, and comparison of their energies under various condition.

### (1) Introduction

We compared the strength of leathers with each other under various thickness as follow.

We took several test-pieces under dry condition.

By tensile testing machine we measured the mechanical properties under its constant pulling speed and we gained the following results.

(2) The expansion theory of the plastic deformation.

If a material has not the plastic deformation, the material which have the deformation  $\xi$  will be satisfied by the following equation having the speed deformation.

$$\frac{d\epsilon}{dt} = \nu = \frac{1}{\mu} \frac{d\tau}{dt}$$

In which,  $\mu$ =modulus of elasticity in the twist. In order to the plastic flow, the speed deformation will be more occurred largely. Assuming that the amount is proportional to the stress, we obtain the next equation.

$$\frac{d\xi}{dt} = \frac{1}{\epsilon} \frac{d\sigma}{dt} + \frac{\sigma}{\zeta} \dots\dots\dots(1)$$

In which,  $\xi$ =percentage of elongation,  $\epsilon$ =modulus of elasticity. This equation is Maxwell's equation. If we more generalize this equation, that is, if we put  $\sigma^n$  for  $\sigma$ , we can gain the following equation.

$$\frac{d\xi}{dt} = \frac{1}{\epsilon} \frac{d\sigma}{dt} + \frac{\sigma^n}{\zeta} \dots\dots\dots(2)$$

When the amplitude of the vibration is very small, we can assume that the deformation  $\xi$  will vibrate with the amplitude a

$$\xi = a e^{i\omega t}$$

In the practical vibration, this equation will be satisfied by the real part.

Similarly  $\sigma = \sigma_0 e^{i\omega t}$

In which,  $\sigma_0$  means the complex number in general.

Then,

$$\begin{aligned} \frac{d\sigma}{dt} &= \sigma_0 i\omega e^{i\omega t} \\ \frac{d\xi}{dt} &= a i\omega e^{i\omega t} \end{aligned}$$

The equation (2) will be became as the next form.

$$ia\omega = \sigma_0 \left\{ \frac{1}{\epsilon} i\omega + \frac{\sigma_0^{n-1}}{\zeta} (e^{i\omega t})^{n-1} \right\} \dots\dots\dots(3)$$

now,  $n=1$

$$\begin{aligned} \zeta_0 &= \left\{ \frac{1}{\epsilon} i\omega + \frac{1}{\zeta} \right\} = \left[ \left\{ \frac{\omega^2}{\epsilon^2} + \frac{1}{\zeta^2} \right\} \epsilon + i \zeta \left\{ \frac{\omega^2}{\epsilon^2} + \frac{1}{\zeta^2} \right\} \right] \dots\dots\dots(4) \\ &= (T' + T''i)a \end{aligned}$$

in which,  $i = \sqrt{-1}$

next,  $n=2$

$$ia\omega = \sigma_0 \left\{ \frac{1}{\epsilon} i\omega + \frac{\sigma_0}{\zeta} e^{i\omega t} \right\} \dots\dots\dots(5)$$

$$\epsilon e^{i\omega t} \sigma_0^2 + i\omega \zeta \sigma_0 - \epsilon \zeta ia\omega = 0$$

$$\therefore \epsilon(\cos\omega t + i\sin\omega t)\sigma_0^2 + i\omega \zeta \sigma_0 - \epsilon \zeta ia\omega = 0$$

$$\therefore \sigma_0 = \frac{-i\omega \zeta \pm \sqrt{-\zeta^2 \omega^2 + 4\epsilon^2 \zeta ia\omega(\cos\omega t + i\sin\omega t)}}{2\epsilon(\cos\omega t + i\sin\omega t)}$$

$$= \frac{i\omega \zeta \pm \sqrt{(-\zeta^2 \omega^2 - 4\epsilon^2 \zeta a\omega \sin\omega t) + i4\epsilon^2 \zeta a\omega \cos\omega t}}{2\epsilon(\cos\omega t + i\sin\omega t)}$$

now, putting as the following form.

$$\sqrt{(-\zeta^2 \omega^2) - 4\epsilon^2 \zeta a\omega \sin\omega t} + i4\epsilon^2 \zeta a\omega \cos\omega t = A + iB$$

we obtain

$$A = \pm \frac{1}{\sqrt{2}} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\epsilon^2 a \sin\omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\epsilon^4 a^2 + 8\zeta \epsilon^2 a \sin\omega t} \}}$$

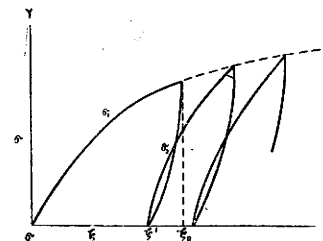


Fig. 1

$$\begin{aligned}
 B &= \pm \frac{1}{\sqrt{2}} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t} \}} \\
 \therefore \sigma_0 &= \frac{-i\zeta \omega \pm \left[ \pm \frac{1}{\sqrt{2}} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t} \}} \right]}{2\varepsilon(\cos \omega t + i \sin \omega t)} \\
 &\quad \pm i \frac{4\varepsilon^2 a \cos \omega t \cdot \zeta \omega}{\sqrt{2} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t} \}}} \dots \dots \dots (6)
 \end{aligned}$$

$$\begin{aligned}
 \therefore \sigma_0 &= \frac{1}{2\varepsilon} \left[ \pm \frac{1}{\sqrt{2}} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t} \}} \cos \omega t \right. \\
 &\quad \left. + \left\{ -\zeta \omega \pm \frac{4\varepsilon^2 a \sqrt{\zeta \omega} \cos \omega t}{\sqrt{-(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t}}} \right\} \sin \omega t \right] \\
 &\quad + i \frac{1}{2\varepsilon} \left[ \left\{ -\zeta \omega \pm \frac{\sqrt{\zeta \omega} \cdot 4\varepsilon^2 a \cos \omega t}{\sqrt{-(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t}}} \right\} \cos \omega t \right. \\
 &\quad \left. \pm \frac{1}{\sqrt{2}} \sqrt{\zeta \omega \{ -(\zeta \omega + 4\varepsilon^2 a \sin \omega t) \pm \sqrt{\zeta^2 \omega^2 + 16\varepsilon^2 a^2 + 8\zeta \omega \varepsilon^2 a \sin \omega t} \}} \sin \omega t \right] \dots \dots \dots (7)
 \end{aligned}$$

$$\begin{aligned}
 &= T' + iT'' \\
 \sigma_0 e^{i\omega t} &= (T' + iT'') \{ \cos \omega t + i \sin \omega t \} \\
 &= aE' \cos \omega t + aiE'' \cos \omega t + iE' \sin \omega t - aE'' \sin \omega t
 \end{aligned}$$

$$\begin{aligned}
 \therefore \sigma &= (T' + iT'') \{ \cos \omega t + i \sin \omega t \} \\
 &= T' \cos \omega t + iT'' \cos \omega t + iT' \sin \omega t - T'' \sin \omega t \\
 &= T' \cos \omega t - T'' \sin \omega t + i(T'' \cos \omega t + T' \sin \omega t)
 \end{aligned}$$

$$\begin{aligned}
 \therefore F(\sigma) &= T' \cos \omega t - T'' \sin \omega t \\
 F(\xi) &= a \cos \omega t
 \end{aligned}$$

$$Q = \int_0^{\frac{2\pi}{\omega}} F(\sigma) \frac{dF(\xi)}{dt} dt = \int_0^{\frac{2\pi}{\omega}} \{ T' \cos \omega t - T'' \sin \omega t \} \sin \omega t dt = \int_0^{\frac{2\pi}{\omega}} (T'' \sin \omega t - T' \cos \omega t) \sin \omega t dt$$

If  $n=1$ ,

$$Q = \int_0^{\frac{2\pi}{\omega}} F(\sigma) \frac{dF(\xi)}{dt} dt = \pi a^2 \frac{\omega}{\zeta} \left\{ \frac{\omega^2}{\varepsilon^2} + \frac{1}{\zeta^2} \right\}$$

In that case, that is, when  $n=2, n=3, \dots$  etc.

It is better to obtain the empirical formula for  $F(\sigma)$ , and we can find the loss energy Q for one cycle, and then it is desired to compare the loss energy of every test-piece.

**(3) The theory of the mechanical properties on leathers.**

In the following equation, when we assume

$$\frac{d\xi}{dt} = \frac{1}{\varepsilon} \frac{d\sigma}{dt} + \frac{\sigma^n}{\zeta}$$

and  $\frac{d\sigma}{dt} = \dot{\xi} \frac{d\sigma}{d\xi} \quad k = \frac{\varepsilon}{\zeta}$

$$\therefore \dot{\xi} = \frac{1}{\varepsilon} \dot{\xi} \frac{d\sigma}{d\xi} + \frac{k\sigma^n}{\varepsilon} = \frac{1}{\varepsilon} \left\{ \dot{\xi} \frac{d\sigma}{d\xi} + k\sigma^n \right\}$$

$$\dot{\xi} \frac{d\sigma}{d\xi} = \varepsilon \dot{\xi} - k\sigma^n$$

$$\frac{d\sigma}{d\xi} = \varepsilon - k \frac{\sigma^n}{\xi}$$

now when,  $n=1$

$$\frac{d\sigma}{d\xi} = \varepsilon - k \frac{\sigma}{\xi}$$

assuming that  $\sigma = 0$  when  $\xi = 0$ ,

$$\begin{aligned} \sigma &= \zeta \dot{\xi} \left( 1 - \zeta \frac{-k\xi}{\dot{\xi}} \right) \\ &= \zeta \dot{\xi} \left[ \frac{k\xi}{\dot{\xi}} - \frac{1}{2} \left( \frac{k\xi}{\dot{\xi}} \right)^2 + \frac{1}{3} \left( \frac{k\xi}{\dot{\xi}} \right)^3 \dots \dots \dots \right] \end{aligned}$$

assuming that  $\dot{\xi} = \text{const}$

and when

$${}^3\sigma = \sigma_0, \quad \xi = \xi_0, \quad \sigma_1 = a\xi + b\xi^2$$

$$-\frac{\dot{\xi}}{k} \log \left\{ \varepsilon - \frac{k\sigma}{\dot{\xi}} \right\} = \xi + \log K$$

$$\log K = -\frac{\dot{\xi}}{k} \log \left\{ \varepsilon - \frac{k\sigma_0}{\dot{\xi}} \right\} - \xi_0$$

$$\therefore \zeta = \xi_0 + \frac{\dot{\xi}}{k} \log \left\{ \frac{\sigma_0 - \frac{\dot{\xi}}{k} \zeta}{\sigma - \frac{\dot{\xi}}{k} \zeta} \right\}$$

$$e^{\frac{k}{\dot{\xi}}(\xi - \xi_0)} = \frac{\sigma_0 - \frac{\dot{\xi}}{k} \zeta}{\sigma - \frac{\dot{\xi}}{k} \zeta}$$

not work

$$\therefore \sigma_2 = \left( \sigma_0 - \frac{\dot{\xi}}{k} \zeta \right) e^{-\frac{k}{\dot{\xi}}(\xi - \xi_0)} + \frac{\dot{\xi}}{k} \zeta \quad \zeta \xi' = \xi_0 + \frac{\dot{\xi}}{k} \log \frac{\sigma_0 - \frac{\dot{\xi}}{k} \zeta}{\sigma - \frac{\dot{\xi}}{k} \zeta}$$

not work

$$\begin{aligned} \Delta w_1 &= \int_0^{\xi_0} \sigma_1 d\xi - \int_0^{\xi_0} \sigma_2 d\xi \\ &= \int_0^{\xi_0} (a\xi + b\xi^2) d\xi - \int_{\xi_1}^{\xi_0} \left\{ \left( \sigma_0 - \frac{\dot{\xi}}{k} \zeta \right) e^{-\frac{k}{\dot{\xi}}(\xi - \xi_0)} + \frac{\dot{\xi}}{k} \zeta \right\} d\xi \\ &= a \frac{\xi_0^2}{2} + b \frac{\xi_0^3}{3} - \left[ \left( \frac{\dot{\xi}}{k} \zeta - \sigma_0 \right) \frac{\dot{\xi}}{k} e^{-\frac{k}{\dot{\xi}}(\xi - \xi_0)} + \frac{\dot{\xi}}{k} \zeta \xi \right]_{\xi_1}^{\xi_0} \\ &= \frac{a}{2} \xi_0^2 + \frac{b}{3} \xi_0^3 - \left( \frac{\dot{\xi}}{k} \zeta - \sigma_0 \right) \frac{\dot{\xi}}{k} + \left( \frac{\dot{\xi}}{k} \zeta - \sigma_0 \right) \frac{\dot{\xi}}{k} e^{-\frac{k}{\dot{\xi}}(\xi' - \xi_0)} - \frac{\dot{\xi}}{k} \zeta \xi_0 + \frac{\dot{\xi}}{k} \zeta \xi' \end{aligned}$$

$$\therefore \text{total work } W \cong \sum \Delta w_1$$

We can compare the strength of leathers by the total work.

**(4) Experiment. in the case,  $n = 1$ .**

We take the following test pieces.

(No. 1) Rollor skin, the leather for picking, length 100mm, width 15mm, thickness 1.9mm, cross sectional area 28.5mm<sup>2</sup>

The following results were shown the deta repeated tensile testing in load and unloading next.

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.397              | 17         | 14.5                 | 2.5                |
| 0.795              | 22.5       | 18.5                 | 4.0                |
| 1.191              | 35.5       | 28.0                 | 5.5                |
| 1.588              | 42.0       | 34.5                 | 7.5                |
| 1.985              | 50.0       | 41.0                 | 9.5                |
| 2.382              | 58.0       | 45.0                 | 13.0               |
| 2.721              | 78.5       | cutting              |                    |

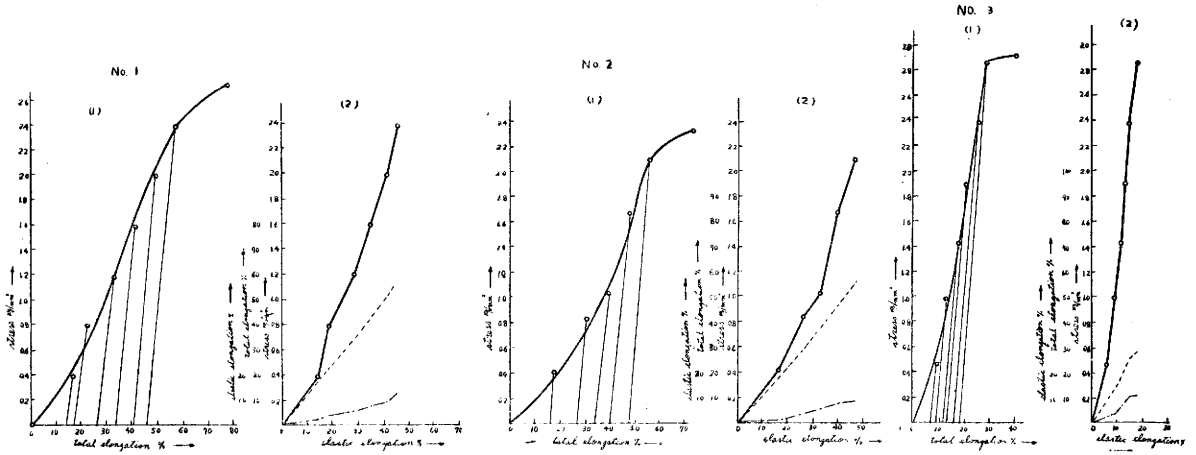


Fig. 2

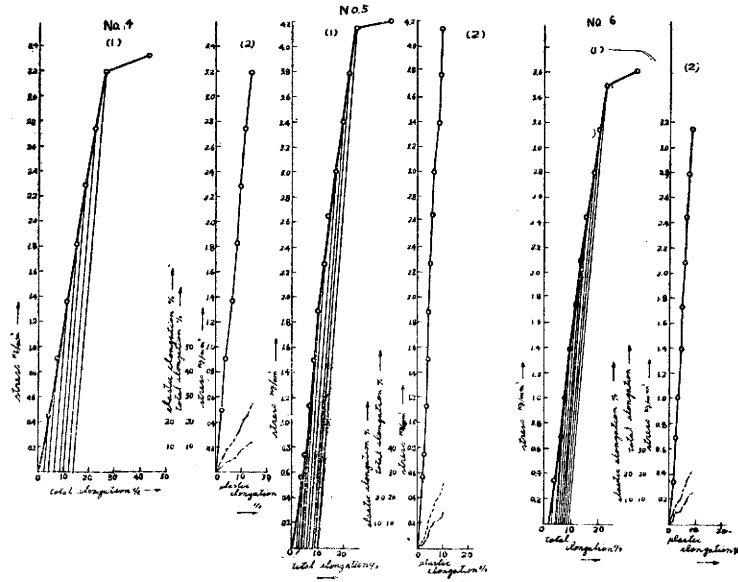


Fig. 3

(No. 2) Roller skin, the leather for picking, length 100mm, width 15mm, thickness 1.8mm, cross sectional area 27mm<sup>2</sup>,

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.420              | 17.5       | 16                   | 1.5                |
| 0.840              | 30.0       | 26                   | 4.0                |
| 1.260              | 39.0       | 33                   | 6.0                |
| 1.680              | 48.0       | 40                   | 8.0                |
| 2.100              | 55.0       | 47                   | 9.0                |
| 2,350              | 73.0       | cutting              |                    |

(No. 3) Roller skin, the leather for picking, length 100mm, width 17mm, cross sectional area 23.8mm<sup>2</sup>.

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.476              | 9          | 6                    | 3.0                |
| 0.952              | 13         | 9                    | 4.0                |
| 1.428              | 18         | 11.5                 | 6.5                |
| 1.904              | 21.5       | 13.0                 | 8.5                |
| 2.380              | 26         | 15.0                 | 11.0               |
| 2.856              | 29         | 18.0                 | 11.0               |
| 2.915              | 41         | cutting              |                    |

(No. 4) Belt skin the leather for picking, length 100mm, width 16mm, thickness 3.1mm, cross sectional area 49.6mm<sup>2</sup>.

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.457              | 4.5        | 2.5                  | 2.0                |
| 0.914              | 8          | 4                    | 4.0                |
| 1.371              | 12         | 7                    | 5.0                |
| 1.828              | 16         | 9                    | 7.0                |
| 2.285              | 19.5       | 10.5                 | 9.0                |
| 2.742              | 23         | 12.5                 | 10.5               |
| 3.199              | 27         | 14.5                 | 12.5               |
| 3.336              | 43.5       | cutting              |                    |

(No. 5) Belt skin, the leather for picking, length 100mm, width 19mm, thickness 3.4mm, cross sectional area 64.6mm<sup>2</sup>,

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.378              | 4          | 2.5                  | 1.5                |
| 0.756              | 5.5        | 3.0                  | 2.5                |
| 1.134              | 7.5        | 4.0                  | 3.5                |
| 1.512              | 9.5        | 4.3                  | 5.2                |
| 1.890              | 12.0       | 5.0                  | 7.0                |
| 2.268              | 13.5       | 5.3                  | 8.2                |
| 2.646              | 15.5       | 6.5                  | 9.0                |
| 3.024              | 18.0       | 7.0                  | 11.0               |
| 3.402              | 20.5       | 9.0                  | 11.5               |
| 3.780              | 23.0       | 10.0                 | 13.0               |
| 4.158              | 25.5       | 10.5                 | 15.0               |
| 4.220              | 39.5       | cutting              |                    |

(No. 6) Belt skin, the leather for picking, length 100mm, width 19mm, thickness 3.4mm, cross sectional area 64.60mm<sup>2</sup>.

| stress             | elongation | permanent elongation | elastic elongation |
|--------------------|------------|----------------------|--------------------|
| kg/mm <sup>2</sup> | %          | %                    | %                  |
| 0.351              | 4          | 2.0                  | 2.0                |
| 0.702              | 7          | 2.2                  | 4.8                |
| 1.053              | 8.5        | 3.5                  | 5.0                |
| 1.404              | 10.5       | 5.0                  | 5.5                |
| 1.755              | 13.0       | 5.5                  | 7.5                |
| 2.106              | 15.0       | 6.5                  | 8.5                |
| 2.457              | 17.0       | 7.0                  | 10.0               |
| 2.808              | 20.0       | 8.0                  | 12.0               |
| 3.159              | 22.0       | 9.5                  | 12.5               |
| 3.510              | 25.0       | cutting              |                    |

### 3 Conclusion,

The experimental results of the stress-strain curve of leathres under the chemical treatments are showed in figures.

Those results are made under the constant pulling speed,  $\dot{\epsilon}$  constant, approximately.

The stress-strain curves for test pieces of No. 1, No. 2, and No. 3 show concave curves but stress-strain curve for test pieces of No. 4, No. 5, No. 6 show approximately straight lines,  $i, e, n=1$  curve. We know that leather would be broken out at a point which strains would be increased suddenly, and we know that the strength for No. 1, No. 2 and No. 3 leather are layer than one for No. 4, No. 5 and No. 6 leathers, because the strength of leathers would be compared with each other by the energy of stress-strain diagrams.

## シアナミド誘導体の液態アンモニアに対する溶解度

大 島 好 文

### Solubility of the cyanamide derivatives in liquid Ammonia.

By Yoshibumi ŌSHIMA

No data being available for solubility of the cyanamide derivatives in liquid ammonia, so the solubility of melamine or dicyandiamide in liquid ammonia was measured at a constant temperature below 30°C in thermostat using specially designed glass pressure bottle.

The solubility of purified melamine (m.p. 352°C) or purified dicyandiamide (m.p. 207°C) in liquid ammonia at 10°C or 20°C was measured. The results were as follows: —

| solvent<br>sample | temp | Soluble amount in 100g liquid ammonia |              |
|-------------------|------|---------------------------------------|--------------|
|                   |      | 10°C ± 0.3°C                          | 20°C ± 0.5°C |
| Dicyandiamide     |      | 104.43 gr                             | 117.46 gr    |
| Melamine          |      | 4.16 gr                               | 3.80 gr      |

### 緒 言

有機化合物の液態アンモニア（以下液安と称す）に対する溶解度を測定した文献は少く、デシアンジアミド及びメラミンに就ても文献が見当たらない。著者は液安加圧下に於てデシアンジアミドを高温に加熱しメラミンを合成する場合オートクレーブ内に於るデシアンジアミドが反応前に液安と共存して如何なる状態にあるか、又反応後に生成メラミンが液安に溶解しているか否かを推定する資料にするためと、広く化学操作に寄与する目的とを以てデシアンジアミド及びメラミンの液安に対する溶解度を測定したので茲に報告する。

### 実験装置及び操作

本実験に使用した装置は普通の恒温槽、標準の液安蒸溜装置並に第1図の如き〔A〕、〔B〕2種のガラス製耐圧容器である。このガラス製耐圧容器の接続部には軟鋼又はデュラルミン製のデョイントを附属せしめる。使用に際してはコックの開閉が自在である如く工夫された真鍮製のコック押へを以てコックの部を締め、内圧によりコックの飛ぶのを防ぎ、予め15気圧までの耐圧試験を実施する。

デシアンジアミド及びメラミンに就て以下に述べる操作を全く同様に行つた。即ち内容約60ccのガラスフィルター付