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HIGHLY LOADED MULTI-STAGE FAN DRIVE TURBINE - PERFORMANCE OF FINAL THREE CONFIGURATIONS

by D. G. Cherry and M. W. Thomas

Prepared by GENERAL ELECTRIC COMPANY Cincinnati, Ohio 45215 for Lewis Research Center



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SUMMARY

This report describes additional experimental results for a program which was originally conducted to investigate aerodynamic means for increasing turbine stage loading and turbine blade loading consistent with high efficiency (Reference 1). Four additional highly loaded fan drive turbines were tested: 1) a threestage turbine using all plain blading (base case), 2) a three-stage turbine with a ten-degree tangentially leaned stator and tandem rotor in stage three, 3) a three-stage turbine using a tandem stator in stage two, and 4) a threestage turbine using a tandem stator in stage two and a ten-degree tangentially leaned stator in stage three. Each turbine was designed to the same velocity diagram, and each used the same constant inside-diameter flowpath.

At design equivalent speed (3169.0 rev/min) and design total-to-total pressure ratio (3.47) the turbine utilizing the stage two tandem stator (with all other bladerows plain) achieved an overall total-to-total efficiency of approximately 0.887 as compared to 0.886 for the all plain blade turbine. Although this represented the highest level of efficiency yet attained for a three-stage turbine in this program, it is well below the level predicted on the basis of previous tests results. Incidence loss on the stage three vane has been identified as the primary cause.

Retest of a two-stage turbine utilizng the stage two tandem stator confirmed the significant increase in two-stage total-to-total efficiency afforded by the use of this bladerow.

INTRODUCTION

The development of high-bypass-ratio turbofan engines for future aircraft propulsion schemes requires the development of fan turbines with increasingly higher work output. The requirements of smaller turbine diameters and reduced rotative speeds generate the need for turbines with higher aerodynamic loadings.

The NASA Highly Loaded Multi-Stage Fan Drive Turbine Program was established to investigate advanced turbine airfoil concepts to meet the requirements of higher loading while maintaining a high level of turbine aerodynamic performance.

During the initial planning of the program seven air turbine configurations were selected for testing which best represented the optimum test plan to evaluate the effects on overall turbine performance of the high lift devices which consisted of tandem stator and rotor airfoils and tangentially leaned stator airfoils.

The seven turbine configurations tested are described in detail in Reference 1. Test results are presented in tabular form in Table VIII of that report.

In view of the configurations that were tested, it became apparent that the testing of additional configurations with the existing airfoil hardware was required in order to completely isolate the individual effects of the tandem and leaned airfoils on the overall performance of the three stage turbine.

The program described herein was a nine-month follow-on investigation to provide additional experimental information on the performance of the existing airfoil bladerows.

The program was divided into three task items of activity. Under Task IA, Testing, five air turbine configurations were assembled and tested. Determination of the overall operating characteristics of these five configurations was accomplished under Task IIA, Data Reduction and Analysis. Task IIIA, Reporting, has as its purpose the orderly presentation of all test results and analyses and is completed with this report.

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TEST VEHICLE

<u>Requirements</u> - The analysis and design of the three fan drive turbines which were investigated are presented in detail in References ³ through 5. An existing highly-loaded fan drive turbine rotating rig was modified for the test and performance phase of the program. The turbine design requirements were scaled for a turbine exit tip diameter of 28.4 inches in order to utilize the existing test rig. The full-size and scaled turbine design requirements are presented below:

Average Pitch Loading, $\frac{gJ\Delta h}{2\Sigma U_p^2}$ 1.51.Equivalent Specific Work, E/θ_{cr} , (Btu/lbm) 33.033.4Equivalent Rotative Speed, $N/\sqrt{\theta_{cr}}$, (rev/min) 20003166Equivalent Weight Flow, $W/\overline{\theta_{cr}}$, (rev/min) 20003166Equivalent Weight Flow, $W/\overline{\theta_{cr}}$, ε/δ , (lbm/sec) 7024Inlet Swirl Angle (degrees)00Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 ≤ 5 Maximum Tip Diameter (inches)45.028.4Number of Stages3 ≤ 5 $W/\overline{T_T}/P_T$ at Inlet108.443.10 $\Delta h/T_T$ 0.06350.0635 $N/\sqrt{T_m}$ 87.7138.96	Parameter	Full Size	Scaled
Equivalent Specific Work, E/θ_{cr} , $(Btu/1bm)$ 33.033.0Equivalent Rotative Speed, $N/\sqrt{\theta_{cr}}$, (rev/min) 20003160Equivalent Weight Flow, W/θ_{cr} ε/δ , $(1bm/sec)$ 7020Inlet Swirl Angle (degrees)000Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 ≤ 5 Maximum Tip Diameter (inches)45.028.4Number of Stages3 ≤ 5 $W/T_T/P_T$ at Inlet108.443.10 $\Delta h/T_T$ 0.06350.0635 $N/\sqrt{T_T}$ 87.7138.96	Average Pitch Loading, $\frac{gJ\Delta h}{2\Sigma U}_{p}^{2}$	1.5	1.5
Equivalent Rotative Speed, $N/\sqrt{\theta}_{cr}$, (rev/min)2000316Equivalent Weight Flow, W/θ_{cr} ε/δ , (1bm/sec)7024Inlet Swirl Angle (degrees)006Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 ≤ 5 ≤ 5 Maximum Tip Diameter (inches)45.028.4Number of Stages3 ≤ 5 ≤ 5 $W/T_T/P_T$ at Inlet108.443.10 $\Delta h/T_T$ 0.06350.0635 $N/\sqrt{T_T}$ 87.7138.98	Equivalent Specific Work, E/θ_{cr} , (Btu/1bm)	33.0	33.0
Equivalent Weight Flow, $W\sqrt{\theta}_{cr} \epsilon/\delta$, (1bm/sec)7024Inlet Swirl Angle (degrees)00Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 ≤ 5 Maximum Tip Diameter (inches)45.028.4Number of Stages3 ≤ 5 $W\sqrt{T_T}/P_T$ at Inlet108.443.10 $\Delta h/T_T$ 0.06350.0635 $N/\sqrt{T_T}$ 87.7138.98	Equivalent Rotative Speed, $N/\sqrt{\theta_{cr}}$, (rev/min)	2000	3169
Inlet Swirl Angle (degrees)0Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 Maximum Tip Diameter (inches)45.0Number of Stages3 $W\sqrt{T_T}/P_T$ at Inlet108.4 $\Delta h/T_T$ 0.0635 $N/\sqrt{T_T}$ $N/\sqrt{T_T}$ 87.7138.98	Equivalent Weight Flow, $W \sqrt{\theta_{cr}} \epsilon / \delta$, (lbm/sec)	70	28
Exit Swirl Angle Without Guide Vanes (degrees) ≤ 5 ≤ 5 Maximum Tip Diameter (inches)45.028.4Number of Stages33 $W\sqrt{T_T}/P_T$ at Inlet108.443.10 $\Delta h/T_T$ 0.06350.0635 $N/\sqrt{T_T}$ 87.7138.96	Inlet Swirl Angle (degrees)	0	0
Maximum Tip Diameter (inches) 45.0 28.4 Number of Stages 3 3 $W\sqrt{T_T}/P_T$ at Inlet 108.4 43.10 $\Delta h/T_T$ 0.0635 0.0635 $N/\sqrt{T_T}$ 87.7 138.98	Exit Swirl Angle Without Guide Vanes (degrees)	<u><</u> 5	<u><</u> 5
Number of Stages 3 $W\sqrt{T_T}/P_T$ at Inlet 108.4 43.10 $\Delta h/T_T$ 0.0635 0.0635 $N/\sqrt{T_T}$ 87.7 138.98	Maximum Tip Diameter (inches)	45.0	28.4
$W\sqrt{T_T}/P_T$ at Inlet 108.4 43.10 $\Delta h/T_T$ 0.0635 0.0635 $N/\sqrt{T_T}$ 87.7 138.98	Number of Stages	3	3
$\Delta h/T_{T}$ 0.0635 0.0635 N/ $\sqrt{T_{T}}$ 87.7 138.98	$W\sqrt{T_T}/P_T$ at Inlet	108.4	43.16
$N/\sqrt{T_{m}}$ 87.7 138.98	Δh/T _T	0.0635	0.0635
\mathbf{T}	N/ $\sqrt{T_T}$	87.7	138.98

<u>Configurations Tested</u> - All bladerows tested in this program were designed to the three-stage turbine velocity diagrams presented in Figure 1. In order to further isolate the effects of the stage two tandem stator, the stage three tangentially leaned stator, and the stage three tandem rotor, a total of five configurations were tested. These configurations are described below.

Configuration	Symbol	Description
1A	PPPPPP	Three-stage turbine with plain blading in all bladerows. This configuration is shown in Figure 2 and is the same as Con-

3

figuration 1 of the original test series. This turbine was run to verify test cell repeatability from the original test series.

Three-stage turbine with leaned stator and tandem rotor in stage three and plain blading in all other bladerows. This configuration was run to investigate the effects of the tandem rotor operating in the improved flow field generated by the leaned stator (see Fig. 88 in Reference 1)

Three-stage turbine with tandem stator in stage two, leaned stator in stage three, and plain blading in all other bladerows. This configuration was tested in anticipation of its representing the optimum combination of bladerows.

Three-stage turbine with tandem stator in stage two and plain blading in all other bladerows. This was an optional configuration (not contractually required) and was tested in order to investigate the effect of the stage two tandem stator operating in a three-stage turbine. In previous testing of a two-stage configuration, this bladerow was shown to be highly efficient relative to the stage two plain stator (see Table VI).

Two-stage turbine with tandem stage two stator and plain blading in all other bladerows. This configuration is shown in Figure 3 and is the same as Configuration 4 of the original test series (Reference 1). This turbine was tested in order to verify the presence of the significant performance payoff realized in the original test series using the tandem stage two stator and was also optional for this program.

Photographs of the turbine blading used in the testing of these five turbine configurations are presented in Figures 4 through 9.

TEST APPARATUS AND INSTRUMENTATION

<u>Test Facility</u> - The five turbine configurations were tested in the same General Electric Company's Evendale Air Turbine Test Facility as the original seven

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2A	 	PPPPLT
	- -	
 3A		PPTPLP

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#2¹

- 4A PPTPPP

- ____
- 5A PPTP

configurations. A typical test facility configuration is shown in Figure 10. A detailed description of the facility as configured for this program is presented in Reference 1.

<u>Data Acquisition System</u> - The data acquisition system for the test facility is capable of recording up to 200 temperatures and 350 pressures, in addition to other specific turbine performance parameters.

Temperature measurements are obtained using Chromel-Alumel thermocouple wire. Corrections for temperature recovery over the expected range of Mach numbers and for flow incidence angles are made in the cell data reduction program.

Pneumatic pressure signals from the turbine rig are fed to precision strain gage pressure transducers within the control room.

For a detailed description of the test cell data acquisition system and calibration techniques, see Reference 1.

<u>Instrumentation</u> - Figure 11 shows the location of the instrumentation used in the testing of the five turbine configurations. An instrumentation scheme was used which permitted removal of downstream turbine stages without requiring the reinstrumentation of upstream stages.

Turbine inlet instrumentation was affixed to the leading edge of the inlet strut frame on each of ten struts located 36 degrees apart, and approximately ten inches upstream of the first stage stator. Turbine inlet temperature was measured with 25 Chromel-Alumel thermocouples mounted in high recovery stagnation tubes affixed to the leading edge of the inlet strut frame on each of five struts 72 degrees apart. They were located radially at the area centers of five equal annular areas. Inlet total pressure was measured with 25 Kiel-type probes located in an identical manner as the total temperatures above, but on five alternate struts 72 degrees apart. These pressures were measured independently by means of the scanner-transducer system and then arithmetically averaged in the data reduction program. They were also pneumatically averaged, using a specially designed averaging block, measuring an average output on a single pressure transducer.

Inlet static pressure was measured with five equally spaced static pressure taps located on both the inner and outer flowpaths in a straight annular section about 1.7 inches upstream of the first stage stator. These static pressure taps were used to check the circumferential uniformity of the flow and to calculate the turbine inlet total pressure.

Interstage static pressures were measured with four static pressure taps installed on both inner and outer flowpath casings, approximately 90 degrees removed, in the cavity area before and after each stator.

Turbine outlet total temperature and total pressure were measured with six fixed circumferential arc rakes 60 degrees apart, located radially at the centers of six equal annular areas, and approximately four inches downstream of the last stage rotor. A total of 36 total temperatures and 72 total pressures were measured. Each rake contained twelve Kiel-type pressure elements located side-by-side, and six shielded thermocouple elements side-by-side. The total pressures were averaged both arithmetically and pneumatically in the same manner as the inlet pressure measurements.

Six turbine outlet static pressures were measured on both the inner and outer flowpaths. Elements were spaced 60 degrees apart and were located approximately four inches downstream of the last stage rotor.

Turbine outlet total temperature and total pressure were additionally measured by a radially and circumferentially traversing combination probe. A fast response pressure differential servo-system aligned the probe with the flow and provided an electrical output proportional to the flow angle. Total temperature, total pressure and flow angle were recorded on X-Y chart recorders as functions of either radial immersion or circumferential position. The probe was located approximately one inch downstream of the last stage rotor.

Air flow to the turbine was measured using a calibrated circular arc venturi which was operated at critical flow conditions. The venturi inlet pressure and temperature were measured using wall static pressure taps and Chromel-Alumel air thermocouple probes located upstream of the venturi throat.

Three independent speed measurements were provided by an indicating system consisting of a 60-tooth gear attached to the turbine shafting and three stationary magnetic sensors located very close to the gear teeth. Electrical impulses resulting from the passing of each tooth provided an electrical frequency proportional to turbine speed. Electrically time integrating this signal provided the speed indication, accurate within +1 rpm. During the course of each data reading, twelve different samples of speed were recorded and arithmetically averaged.

The torque measurement system consisted of a dual bridged shaft-mounted torque sensor. The strain sensitive spool section was located between the turbine shaft and the waterbrake shaft with a specially designed slip ring mounted behind the waterbrake to transmit electrical signals to the digital recorder. Each bridge was excited with its own independent electronics system and read out or displayed through the digital data acquisition system.

Torque calibrations were performed in place using a precision torque arm and dead weights, whose weight values are traceable to the National Bureau of Standards. Dead weight calibrations were conducted prior to each test run to verify repeatability of torque zeros and bridge linearity. In addition, extensive temperature calibrations were made to define torque zero and modulus changes over the operational temperature range, even though these effects are less than 0.25 percent.

TEST PROCEDURE

The turbine inlet conditions were set at 700° R and 30 psia at all test points.

The performance mapping of the turbine was accomplished by selecting test points within the following range of variables:

- Speed from 80 to 120 percent of design speed
- Pressure ratio from that corresponding to approximately 75 percent design ideal enthalpy drop to a pressure ratio corresponding to approximately 105 percent design ideal enthalpy drop except configuration 4A which was not tested below a pressure ratio corresponding to approximately 93 percent of design ideal enthalpy drop.

The following performance data were obtained at each test point:

- Turbine weight flow
- Rotative speed
- Torque
- Inlet total temperature
- Inlet total and static pressures
- Exit absolute flow angles
- Exit total and static pressures
- Exit total temperatures
- Flowpath hub and tip interstage static pressures

Three complete sets of data were recorded at each test point and processed through the on-line computer which permitted an immediate evaluation of the reduced data.

Key performance parameters were continually monitored to insure accuracy and consistency of the test data. The design point was periodically reset throughout the testing to monitor the repeatability of the facility and the design point calculations.

One radial and three circumferential traverses were made at each test point to record the turbine exit total pressure, total temperature and absolute flow angle. The circumferential traverses were taken at 10, 50, and 90 percent of the last stage rotor blade height.

A detailed rotor exit survey was made at the design speed and design pressure ratio for each of the four three-stage turbine configurations tested. The survey for each configuration included seven circumferential traverses of total temperature, total pressure, and flow angle at the radial centers of seven equal annular areas. The traverses encompassed at least two last stage stator wakes.

DATA REDUCTION PROCEDURE

Overall Performance - Two calculation schemes were used to reduce the overall performance data. The two methods differed in only one respect. The preliminary test cell data reduction program used measured exit total pressures for all performance calculations while the final data reduction was performed using calculated exit total pressure. This exit total pressure was calculated using continuity by determining an integrated average flow angle from the traverses and combining it with the exit total temperature based on measured torque and the average of measured exit hub and tip static pressures.

A more detailed description of all the calculation procedures used in the data reduction may be found in Appendix A.

The following overall performance parameters were calculated for each of the three readings taken at each test point.

- 1. Calculated total-to-total pressure ratio as obtained from indirect measurement.
- 2. Calculated total-to-static pressure ratio as obtained from indirect measurement.
- 3. Equivalent speed.
- 4. Equivalent weight flow.
- 5. Equivalent weight flow-speed parameter (product of equivalent speed and weight flow).
- 6. Equivalent torque
- 7. Equivalent specific work
- 8. Ideal equivalent specific work
- 9. Efficiency (total-to-total).
- 10. Blade-jet speed ratio

These parameters are presented in Tables I through V for turbine configurations 1A through 5A respectively.

<u>Stage Performance</u> - Calculations were performed to determine the efficiency of each stage of the various turbine configurations when the three stage turbine was operating at its design speed and design total-to-total pressure ratio. Design total-to-total pressure ratio for the three stage plain blade turbine (Configuration 1A) was defined to be that at which the design equivalent specific work of 33.0 Btu/lbm was extracted. All stage efficiency calculations were performed with a three-stage turbine total-to-total pressure ratio of 3.47. In order to determine the stage efficiencies, it was necessary to determine the key performance parameters of the two-stage and one-stage turbine when the three-stage turbine was operating at its design point. Basic to the stage efficiency calculation was the assumption that removal of downstream turbine stages did not alter the design point performance of the two-stage and onestage turbines, e.g., the two-stage turbine behaved identically when run by itself and when run in the three-stage turbine.

A detailed outline of the stage efficiency calculation along with a sample calculation is presented in Appendix B.

<u>Rotor Exit Survey Calculations</u> - The rotor exit surveys of total pressure, total temperature, and absolute flow angle, which were taken at the design point of each turbine configuration, were used to construct contour plots of local efficiency and local absolute flow angle. Local efficiencies were calculated from the following parameters:

- Measured inlet total temperature
- Calculated inlet total pressure based on continuity using measured inlet static pressure and measured airflow
- Local exit total pressure measured by the traverse probe
- Local exit total temperature measured by the traverse probe

EXPERIMENTAL RESULTS AND DISCUSSION

Test Cell Repeatability - In order to verify the consistency of air turbing test facility data acquired during this test series with that acquired during the original test series, the three-stage turbine with all plain bladerows was rerun as a base case. Figures 12 through 14 present design speed curves of equivalent torque, equivalent weight flow, and turbine total-to-total efficiency versus turbine total-to-total pressure ratio for Configuration 1A (PPPPPP) of this series and Configuration 1 (also PPPPPP) of the original series. These plots confirm the test cell repeatability, thus establishing a base for comparison between the original turbine test series and this follow-on test series.

<u>Overall Performance</u> - The reduced data and calculated parameters are presented in the following curves for each turbine configuration:

- 1. Equivalent torque versus calculated total-to-total pressure ratio.
- 2. Equivalent weight flow versus calculated total-to-total pressure ratio.
- 3. Equivalent specific work versus calculated total-to-total pressure ratio.
- 4. Total-to-total efficiency versus calculated total-to-total pressure ratio.
- 5. Total-to-total efficiency versus blade-jet speed ratio.
- 6. Equivalent specific work versus equivalent weight flow speed parameter with lines of constant calculated total-to-total pressure ratio, constant speed, and constant efficiency.

The above curves utilize constant values of equivalent speed as a parameter and are shown in Figures 15 through 43.

In Figures 44 through 47, some of the reduced data for the plain blade turbine build (Configuration 1A) are compared to the pretest predictions which were originally presented in Reference 3. The data show reasonable agreement with predictions in the vicinity of the design point, with some divergence occurring at far off-design points. The predictions were made with the use of an off-design turbine computer program (Reference 6) and some disagreement was expected because of the assumptions used in the program. The computer program uses constant loss coefficients (such as bladerow efficiencies and rotor and stator total pressure recovery factors) at each operating point. The differences seen in the equivalent weight flow versus pressure ratio curves was attributed partially to the coefficients used in the computer program, and partially to variations in bladerow throat areas in the assembled hardware compared to design intent.

In Figure 48, total-to-total efficiency versus total-to-total pressure ratio for the design equivalent speed line is compared for all three-stage turbine configurations. At the design point (Pressure ratio = 3.47 for Configuration 1A) the efficiencies fell within four-tenths of one-percent of each other. Configuration 2A (PPPPLT) exhibited the lowest design point efficiency of all the three-stage builds, due primarily to a lower bladerow efficiency for the stage three tandem rotor (see Stage Performance). Configuration 3A (PPTPLP) showed no significant increase in design point efficiency over the base case (Configuration 1A-PPPPPP), and Configuration 4A (PPTPPP) demonstrated an advantage of less than one-tenth of one-percent in design point efficiency over the base case. While this represents the highest level of efficiency yet attained in this program, it is well below the full one-percent increase predicted for Configuration 4A on the basis of test results from a two-stage turbine utilizing the stage two tandem stator (Table VIII, Reference 1). The discrepancy between the expected level of performance and the level realized during actual test can be partially explained by the fact that a high stage exit swirl from the two-stage turbine with a tandem stage two stator (see Rotor Exit Survey, this report) resulted in excessive positive incidence on the stage three stator, a bladerow which is characterized by its extreme sensitivity to positive incidence. A detailed analysis of the losses associated with the higher positive incidence is presented in Appendix C. Results of that analysis indicate that a loss of approximately one-half of one-percent in three-stage turbine efficiency is attributable to excessive positive incidence on the stage three stator. Thus most of the benefit derived from the improved performance of the tandem stage two is masked in the three-stage build by a poorly performing stage three. The fact that Configuration 3A (PPTPLP) performance is below that of Configuration 4A seems to indicate that the incidence problem is accentuated somewhat by stator lean.

In Figure 49, equivalent weight flow versus total-to-total pressure ratio for the design equivalent speed line is compared for all three-stage configurations. Note that the equivalent weight flow for the configurations utilizing the tandem stage two stator is lower than that for the configurations utilizing the plain stator in stage two. This difference in flow was also noticed during the original test series and is reported in Reference 1. Figures 50a and 50b present total-to-total efficiency versus total-to-total pressure ratio for the two-stage turbines. Figure 50a compares efficiency based on measured total temperature drop across the turbine and measured inlet and exit total pressures for Configuration 4 (PPTP) and Configuration 2 (PPPP), both from the original test series. This is included to further substantiate the improved performance of the two-stage tandem turbine which was originally reported in Reference 1. Figure 50b compares efficiency based on measured torque and calculated inlet and exit total pressures for Configuration 5A (PPTP) and Configuration 2 (PPPP). Again, the advantage afforded by the tandem stator is obvious.

In Figures 51 through 54, curves of static pressure normalized by inlet total pressure versus axial station are presented for various turbine pressure ratios to illustrate the interstage hub and tip static pressure behavior of the 3-stage turbine configurations. Figure 51 (Configuration 1A - PPPPPP) indicates that the stage one rotor hub at lower pressure ratios had positive reaction and as pressure ratio increased, the reaction became negative. Stage one was designed for approximately eight percent positive hub reaction, while test data indicated slightly negative hub reaction at the design point. Figure 51 also indicates that the stage three rotor hub at lower pressure ratios had positive reaction which became negative reaction as the pressure ratio increased. In this case, the stage three rotor hub was designed for approximately twenty percent negative reaction. Figures 52 and 53, normalized static pressure for Configurations 2A (PPPPLT) and 3A (PPTPLP) respectively, illustrate the influence of the stage three tangentially leaned stator on reaction. Both of these leaned stator configurations had a positive reaction stage three rotor throughout their entire operating range.

<u>Stage Performance</u> - Stage performance calculations were performed to evaluate the performance of the all-plain third stage in Configuration 4A (PPTPPP) and of the leaned/tandem (/LT) third stage in Configuration 2A (PPPPLT). 11

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In Appendix B of Reference 1, the stage efficiency of the all-plain third stage (/PP) was calculated to be 0.923 while operating in Configuration 1 (PPPPPP). In Appendix B of this report, however, the efficiency of that same state was calculated to be only 0.877 while operating in Configuration 4A (PPTPPP). The major part of this third stage performance decrement has been attributed to the positive incidence problem, previously discussed, that arises when the stage two tandem stator is used in the three-stage builds. The reader is again referred to Appendix C of this report for a more detailed analysis of the incidence loss problem.

The combination of the stage three tangentially leaned stator and stage three tandem rotor was calculated to have a stage efficiency of 0.909 while operating in Configuration 2A (PPPPLT), while the combination of the stage three plain stator and the stage three tandem rotor exhibited a stage efficiency of 0.918 operating in Configuration 5 (PPPPT) of the original test series. Stage three efficiency attained utilizing all plain blading was 0.923. These results indicate that the stage three tandem rotor is inherently less efficient than the stage three plain rotor, even when operating in the improved pressure field generated by the leaned stator. Results of stage performance calculations for this program and for the original program are summarized in Table VI of this report.

<u>Rotor Exit Survey</u> - Turbine efficiency contour plots showing local efficiency as a function of radius ratio and circumferential position for each turbine configuration design point are presented in Figures 55 through 58. These plots are useful for observing trends in so far as they indicate the regions of high efficiency at the pitchline between the last stage stator wakes and the regions of low efficiency in the vicinity of the tip, with a large decrease in efficiency toward the hub.

The temperature and pressure data used to construct these plots were manually read from the X-Y charts produced by the traversing survey probe. The accuracy of this technique is only sufficient to determine local trends and not absolute level of local efficiency; thus, the reader is cautioned against drawing conclusions about the relative performance of the various turbine configurations from these contour plots.

Figures 59 through 64 present contour plots showing local exit swirl angle as a function of radius ratio for each turbine configuration design point. The distinguishing characteristic among the three-stage turbines is the difference in swirl gradient from hub to tip for those turbines utilizing the stage three tangentially leaned stator (Configuration 2A-PPPPLT, and Configuration 3A-PPTPLP) as opposed to those utilizing the plain stator (Configuration 1A-PPPPPPP, and Configuration 4A-PPTPPP). The stator lean tends to bring the hub and tip swirls closer to the pitch value. This trend was also reported in Reference 1.

The swirl contour for Configuration 2 (PPPP) of the original test series is included as Figure 64 to provide a comparison with Configuration 5A (PPTP), shown in Figure 63. This comparison clearly illustrates the increased level of swirl for the two-stage tandem turbine.

<u>Recommended Improvements</u> - The results of this follow-on series of air turbine tests together with the test results from the original program (see Reference 1) suggest the following areas of potential improvement in three-stage turbine performance:

1. Stator Redesign

- a) Redesign the stage two and stage three stators for slightly negative incidence as indicated by the results of the rotor exit survey and cascade test results (see Reference 2). The anticipated improvement in three-stage turbine design point efficiency resulting from such a redesign is one-half of one-percent (see Appendix C).
- b) Investigate a tandem arrangement for the stage three stator using the same solidity as the stage three plain stator.
- c) Redesign all stators using a curvilinear lean distribution, with positive lean at the hub and negative lean at the tip. Figure 65 presents the radial efficiency profiles for Configuration 1

(PPPPPP) and Configuration 7 (PPPPLP) of the original program. This comparison illustrates the improved performance in the hub region realized by using a stator with constant 10° positive tangential lean (see Figure 6). Note, however, that a definite performance penalty was incurred at the tip. Similar results were noted in Reference 7, where stators with curvilinear lean reduced losses significantly in annular cascades with sloped outer walls.

2. Rotor Redesign

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Redesign all rotors for slightly negative incidence to provide a high level of performance at both design and off-design operating conditions. Cascade test results reported in Reference 2 indicate a high sensitivity to angle of attack.

3. Non-Free Vortex Velocity Diagram

Establish a radial work distribution to extract more work in the high performance pitch region and to unload the hub and tip regions. This would effectively decamber the bladerows near the endwalls, resulting in lower secondary losses in these regions. The radial efficiency profiles in Figure 65 provide some indication of the need to reduce the strong endwall secondary flow fields.

4. Redesign the Three-Stage Turbine to Include Outlet Guide Vanes (0.G.V.'s)

Addition of O.G.V.'s to the three-stage turbine would allow a more highly loaded third-stage, resulting in a more uniform stage energy split and a positive reaction stage three rotor, while keeping turbine exit swirl within desired limits. Reference 8 reports the test results for a very highly loaded 4-1/2-stage turbine in which the use of O.G.V.'s resulted in a loss of approximately one-half of one-percent in measured total-to-total efficiency relative to a four-stage configuration without O.G.V.'s. The concept of a 3-1/2-stage turbine involves a tradeoff between diffusion losses in the O.G.V.'s and the anticipated advantages to be gained from redistribution of energy splits. A parametric study incorporating the experimental results of Reference 8 into several different 3-1/2-stage turbine velocity diagrams is suggested to determine the practicality of such a design.

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MECHANICAL EVALUATION

The plain and tandem rotor blades were vibration and fatigue tested as part of the original program in order to insure their mechanical integrity during test.

The vibration analysis consisted of bench testing to confirm analytically established natural frequencies and node patterns (see References 3 and 4) for the plain and tandem airfoils.

Bench fatigue endurance testing was carried out to isolate possible failure regions and corresponding stress levels.

Results of this testing are presented in Reference 1 and indicated that the blade rows possessed sufficient mechanical integrity for successful operation in the air turbine.

SUMMARY OF RESULTS

Four highly loaded fan drive turbines were tested: (1) a three-stage turbine using all plain blading (base case), (2) a three-stage turbine with a ten-degree tangentially leaned stator and a tandem rotor in stage three, (3) a three-stage turbine using a tandem stator in stage two, and (4) a three stage turbine using a tandem stator in stage two and a ten-degree tangentially lean stator in stage three. Each turbine was designed for the same velocity diagram and each used the same flowpath. The most significant results of the testing and evaluation are summarized below:

- 1. At the design speed and pressure ratio $(P_{T_0}/P_{T_3} = 3.47, N/\sqrt{\theta_{cr}} = 3169.0)$ the plain blade turbine (Configuration 1A - PPPPPP) achieved an overall total-to-total efficiency of 0.886.
- 2. The significant increase in design point total-to-total efficiency which was predicted for the tandem turbines (Configuration 3A -PPTPLP and Configuration 4A - PPTPP) on the basis of previous testing of the stage two tandem stator did not materialize during test. Excessive positve incidence on the stage three stator in these configurations has been identified as the primary cause. Configuration 4A (PPTPPP) did, however, exhibit a design point efficiency of approximately 0.887, the highest level of performance yet attained in this program.
- 3. The use of a stage three tandem rotor in Configuation 2A (PPPPLT) resulted in a penalty of approximately two-tenths of one percent in total-to-total efficiency.
- 4. Retest of the two-stage tandem turbine (Configuration 5A-PPTP) confirmed the significant increase in two-stage turbine total-to-total efficiency afforded by the use of a tandem stator in stage two.

APPENDIX A

OVERALL PERFORMANCE CALCULATION

Flow Angle - In order to evaluate turbine performance on the basis of turbine exit total pressure calculated from continuity, an average turbine exit flow angle was determined. The turbine exit flowpath was divided into streamtubes, and measured values of swirl angles, total pressure, and total temperature were used to satisfy continuity within each streamtube. The turbine exit measured static pressure was assumed to vary linearly from hub to tip. The determination of the average turbine exit flow angle processed as follows:

where:

$$\rho_{i} V_{i} = P_{S_{i}} \sqrt{\frac{\gamma g}{RT_{i}}} \sqrt{\frac{2}{\gamma - 1} \left[\left(\frac{P_{T}}{P_{S}} \right)_{i}^{\gamma - 1} \right]} \sqrt{\left(\frac{P_{T}}{P_{S}} \right)_{i}^{\gamma - 1}}$$

 P_{TT} = Measured total pressure at center of i-th streamtube.

P_S = Static pressure at center of i-th streamtube based on linear variation in measured static pressure from hub to tip

$$T_{T}$$
 = Measured total temperature at center of i-th streamtube

- Γ = Swirl angle
- ρ = Density
- V = Absolute velocity

A = Area

- m = Number of streamtubes
- i = Subscript denoting streamtube value

ann = Subscript denoting average value for total annulus

The average velocity representing the turbine exit flow field was calculated by conserving the axial and tangential components of momentum, such that

V z_{avg}

V_i

= Tangential component of absolute velocity ٧., V_z = Axial component of absolute velocity = Weight flow through i-th streamtube = $\rho_i V_i A_i \cos \Gamma_i$ W,

/ m Σ·W i=1

 $\left(\frac{P_{S}}{P_{T}}\right)^{\gamma-1}$

The average turbine exit total temperature was determined through an energy balance of the annular streamtubes.

> $= \begin{pmatrix} m & m \\ \Sigma & W_{i} & T_{T_{i}} \end{pmatrix} / \begin{pmatrix} m & m \\ \Sigma & W_{i} \\ i = 1 \end{pmatrix}$ T T_{avg}

 $V_{avg} = \left(V_{u_{avg}}^{2} + V_{z_{avg}}^{2} \right)^{1/2}$ $V_{u_{avg}} = \left(\frac{\substack{m \\ \Sigma W_{i} V_{i} \sin \Gamma_{i}}}{\substack{i=1 \\ i=1}} \right) / \frac{\substack{m \\ \Sigma W_{i} \sin \Gamma_{i}}}{\substack{i=1 \\ i=1}}$

 $\begin{pmatrix} m \\ \Sigma W_i V_i \cos \Gamma_i \\ i=1 & i & i \end{pmatrix}$

= $\int 2g Jc_p T_{T_i} \left| 1 - \right|$

The average density at the turbine exit was obtained from the equation of state.

$$\rho_{avg} = \frac{\frac{P_{S}}{avg}}{\frac{R}{R} \frac{T_{S}}{S}}$$

where

$$T_{S_{avg}} = T_{T_{avg}} - \frac{v^2}{\frac{avg}{2g_{Jc_{p}}}}$$

Overall Performance - After obtaining the average turbine exit flow angle, the exit total pressure was calculated in the following manner:

$$P_{T_3} = P_{S_3} \left(1 + \frac{\gamma - 1}{2} M_3^2\right)^{\gamma/\gamma - 1}$$

Turbine exit Mach number, M_3 , was determined from the following relationship:

$$\frac{\sqrt[W]{R T_{T_3}}}{\frac{P_S A_{ann} \cos \Gamma_{avg}}{P_s M_{ann}}} = \sqrt{\gamma g} M_3 \sqrt{1 + \frac{\gamma - 1}{2} M_3^2}$$

Turbine exit total temperature, T_{T_3} , was determined as follows:

$$T_{T_3} = T_{T_{\infty}} - \frac{\Delta h}{c_p}$$

where

$$\Delta h = \frac{2\pi NT}{60 JW}$$

2π Ντ

N = Turbine rotative speed, rev/min
τ = Measured torque, ft-lbf
T_{T₀₀} = Measured turbine inlet total temperature, ° R
W = Measured turbine weight flow, lbm/sec

Turbine inlet total pressure was calculated in the same manner as the turbine exit total pressure. The calculation used measured airflow, measured inlet total temperature, the average of measured hub and tip static pressures, and the assumption of zero inlet swirl angle.

The remaining parameters used in the overall performance calculation were obtained as follows:

 $\delta = P_{T_0}/14.696$ $\theta_{cr} = T_{T_{00}}/518.688$ $\epsilon = 1.0 \text{ (for } \gamma = 1.4\text{)}$ Equivalent Speed, N EQV = N/ $\sqrt{\theta_{cr}}$

cr Equivalent Weight Flow, WA EQV = $W\sqrt{\theta}_{cr} \varepsilon/\delta$

Weight Flow-Speed Parameter, WAN EQV = $WN\varepsilon/60\delta$

Equivalent Torque, TQ EOV = $\tau \epsilon / \delta$ Equivalent Specific Work, DH EQV = $\frac{E}{\theta_{cr}} = \frac{2\pi N\tau}{60 J \theta_{cr}} W$

Ideal Equivalent Specific Work, DHI EQV =

$$\left(\frac{E}{\theta_{cr}}\right)_{ideal} = c_{p} T_{T_{oo}} \left[1 - \left(\frac{P_{T_{3}}}{P_{T_{o}}}\right)^{\frac{\gamma-1}{\gamma}}\right] / \theta_{cr}$$

Total-to-total Efficiency, ETA TT =

$$n_{TT} = \left(\frac{E}{\theta_{cr}}\right) / \left(\frac{E}{\theta_{cr}}\right)_{ideal}$$

Blade-Jet Speed Ratio, U/CO =

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$$v = \left\{ \frac{KN^2}{c_p T_{T_{00}} \left[1 - \left(\frac{P_{S_3}}{P_{T_0}} \right)^{\frac{\gamma-1}{\gamma}} \right]} \right\}^{1/2}$$

$$K = \sum_{i=1}^{m} \left(\frac{\pi D_{P_i}}{720} \right)^2 / 2g J$$

where:

where:

m = number of turbine stages

D = pitchline diameter of the i-th rotor

APPENDIX B

STAGE EFFICIENCY CALCULATION

Calculations were performed to determine the efficiency of the third stage of Configuration 2A (PPPPLT) and of Configuration 4A (PPTPPP) with both three stage turbines operating at the design point. In order to compare the stage efficiencies on an equal basis, calculations were performed for a three-stage turbine total-to-total pressure ratio of 3.47. This is the pressure ratio at which the design equivalent specific work of 33.0 Btu/lbm is extracted when the three-stage plain blade turbine operates at the design equivalent speed.

The calculation procedure is outlined below:

- 1. Enter curves of equivalent specific work versus total-to-total pressure ratio at design equivalent speed for the three-stage turbines to obtain equivalent specific work at a pressure ratio of 3.47.
- Enter three-stage turbine curves of normalized static pressure versus total-to-total pressure ratio at a pressure ratio of 3.47 to determine normalized static pressure at the hub and tip of stage two exit.
- 3. At the stage two normalized hub and tip exit static pressures, enter curves of normalized static pressure versus total-to-total pressure ratios across the two-stage turbines.
- 4. Enter curves of equivalent specific work versus total-to-total pressure ratio for the two-stage turbine to determine its equivalent specific work.
- 5. Using the above information and Keenan and Kaye's <u>Gas Tables</u> (Reference 9), calculate the stage efficiencies.

The following example shows how the efficiency of the all plain third stage (/PP) operating behind the two-stage tandem combination (PP/TP) was calculated using test results for configuration 4A (PPTPPP) and Configuration 5A (PPTP).

1. At $(P_{T_0}/P_{T_3})_{4A} = 3.47$, $(E/\theta_{cr})_{4A} = 33.05$ Btu/lbm. 2. At Stage Two exit, $P_c/P_{T_0} = 0.300$

- 3. For the two-stage turbine, $(P_{T_0}/P_{T_{1.5}})_{5A} = 2.66$
- 4. For the two-stage turbine, $(E/\theta_{cr})_{5A} = 26.78$
- 5. Stage efficiencies are calculated from the above information and the accompanying sketch which was constructed using Table 1 of Reference 9.

Configuration 4A	E/0 _{cr}	Δh
Stage 1 & Stage 2	26.78	36.141
Stage 3	6.27	8.462
Total	33.05	44.603



Stage Three Efficiency Calculation

$$n_{TT} = \frac{h_2 - h_3}{h_2 - h_{31}} = \frac{131.419 - 122.957}{131.419 - 121.773} = .877$$

Similar calculations for Configuration 2A (PPPPLT) yield the following results:

Configuration 3A	E/θ cr	Δh
Stage 1 + Stage 2	26.38	35.601
Stage 3	6.52	8.799
Total	32.90	44.400

$${}^{n}TT = \frac{{}^{n}T1.5 - {}^{n}T3}{{}^{h}T_{1.5} - {}^{h}T_{3i}} = \frac{131.959 - 123.16}{131.959 - 122.276} = .909$$

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These results have been incorporated into Table VIII of Reference 1 which is presented as Table VI of this report.

APPENDIX C

ANALYSIS OF INCIDENCE LOSSES

As a result of the significant increase in performance of the two-stage tandem turbine (PPTP) over the two-stage plain turbine (PPPP) which was noted during the original test series, it became highly desirable to test the stage two tandem stator in a three-stage build. Based on stage performance analyses a full one-percent increase in design point total-to-total efficiency was expected. During actual test of this turbine (Configuration 4A-PPTPPPP), however, less than one-tenth of one-percent increase in efficiency was realized (see Figure 48).

Data reduction and analysis have revealed that excessive positive incidence on the stage three plain stator accounted for approximately one-half of onepercent of this performance decrement. A review of that analysis is presented below.

Figures 66 and 67 present contour plots showing incidence of the stage three stator as a function of radius ratio and percent circumferential location. Figure 66 shows that, at the design point, a typical two-stage tandem turbine (PPTP) produces an average of about five and one-half degrees positive incidence on the stage three vane, while Figure 67 shows an average of about three degrees for the two-stage plain turbine (PPPP).

Figure 68 presents a plot of vane cascade efficiency, n_v , versus incidence angle, i, for a typical plain stator as cross-plotted from cascade data in Reference 2. Note the sharp drop off in n_v with increasingly positive incidence, a result of suction side separation.

Using Figures 66, 67, and 68, the incidence angle, i, and stage three stator cascase efficiency, n_{v3} , can be tabulated as follows:

Configuration	(i) Stage 3	ⁿ v3
PPPPPP	3.0°	.9580
PPTPPP	5.5°	.9375

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A velocity diagram study was conducted using Reference 6 to determine the derivative of three-stage turbine efficiency, n_{TT} , with respect to stage three vanes efficiency, n_{v3} . Results of this study indicate that, for a one-percent change in stage three vane efficiency, a resulting change of one-quarter of one-percent in three-stage turbine efficiency would occur.

Applying this efficiency derivative to the values of n_{v3} , in the table, the penalty in three-stage turbine efficiency due to excessive positive incidence can be calculated as follows:

$$\Delta \eta_{\rm TT} = \frac{\partial \eta_{\rm TT}}{\partial \eta_{\rm V3}} \quad \Delta \eta_{\rm V3} = (.25)(.9375 - .9580) = -.0051$$

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The results of this analysis, therefore, indicate that a loss of one-half of one percent in three-stage turbine total-to-total efficiency is attributable to incidence loss.

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APPENDIX D

LIST OF SYMBOLS

A	Area (in. ²)
с _р	Specific heat at constant pressure (ft ² /sec ² °R)
D	Diameter (in.)
do	Bladerow throat dimension (in.)
∆h	Turbine energy extraction (Btu/lbm)
^{∆h} stg	Stage energy extraction (Btu/lbm)
h _{ex}	Height of bladerow at exit (in.)
^h th	Height of bladerow at throat (in.)
i	Incidence angle (degrees)
L	Tangetially leaned bladerow
М	Mach number
m	Number of bladerows, streamtubes, or stages
N	Rotational speed (rev/min)
n	Number of vanes or blades
P	Plain bladerow
P _s	Static pressure (psia)
Ps3	Turbine exit static pressure
^Р т	Total pressure (psia)
P _{To}	Turbine inlet total pressure
P _{T3}	Turbine exit total pressure
R	Gas constant (ft ² /sec ² °R)
Т	Tandem bladerow
т _s	Static temperature (°R)

т _т	Total temperature (°R)
T _{Too}	Turbine inlet total temperature
T _{T3}	Turbine exit total temperature
t	Spacing (in.)
U	Wheel speed (ft/sec)
v	Absolute velocity (ft/sec)
W	Mass flow rate (lbm/sec)
E/0 _{cr}	Equivalent specific work (Btu/lbm)
W√0cr ^{ε/δ}	Equivalent weight flow (1bm/sec)
$N/\sqrt{\theta_{cr}}$	Equivalent rotative speed (rev/min)
₩Nε/60δ	Weight flow - speed parameter (lbm/sec ²)
gJ∆h/2U ²	Loading factor
°o	Vane inlet absolute flow angle (degrees)
α ₁	Vane exit absolute flow angle (degrees)
^β 1	Blade inlet relative flow angle (degrees)
^β 2	Blade exit relative flow angle (degrees)
Γ	Stage leaving swirl angle (degrees)
Ŷ	Specific heat ratio
δ	Ratio of turbine pressure to pressure at standard sea level conditions
ε	Function of γ defined as $\frac{\gamma_{SL}}{\gamma} \left[\left(\frac{\gamma+1}{2} \right)^{\gamma/\gamma-1} / \left(\frac{\gamma_{SL}+1}{2} \right)^{\gamma} \right]^{\gamma}$ SL SL
ⁿ TT	Total-to-total efficiency based on measured torque and calculated inlet and exit total pressures.
ⁿ TT	Total-to-total efficiency based on measured temperature drop and measured inlet and exit total pressures.
n v	Cascade efficiency

θ Squared ratio of critical velocity at turbine inlet temperature cr to critical velocity at standard sea level temperature

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μ	Viscosity (1bm/sec-ft)
ν	Blade-jet speed ratio
ρ	Density (lbm/ft ³)
τ	Torque (ft-1bf)
^T eq	Equivalent torque (ft-lbf), $\tau_{eq} = \tau \epsilon / \delta$

Subscripts

h	Hub
i	Current axial station, stage, streamtube, or ideal
р	Pitch
r	Radial component
t	Tip
u	Tangential component
Z	Axial component

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. (qqqqq	FLOWANG	6 8.87 8.87	6 9,14	0 13.58	5 13.72	6 13,64	0 0 25 0		4 11.03	5 11,20	4 11,17		6 14.60	5 9,00	8 8,82	4 8.75	999 90 10 10 10 10 10 10 10 10 10 10 10 10 10	29°8		9.29	8 9.32	6 19 . 38	8 19.38		40.01 0	8 14.26	3 9,21	1 9.06		1 11.26	0 11.19	4 8,78	2 8 75	10.10	5 11.85	3 13.90	1 24.77	0 24,80	7 24,79	4 19.67		1 10.02	16°6 0
la (p	U/C0	0.368 0.368	0.369	0.445	2443°0	7#7*0	0.406	90400	0.332	0.332	0.332	462.0	0.296	0.370	0.370	0.370	0.370		0 / C * 0	1910	0,382	0.463	0.463		0.424	0.423	0,346	976		0.308	0.308	0.370	0/5-0		0.402	0.402	0.483	0.483	0.482			191.0	0,363
ration	ETA TT	0.8856 0.8856	0.8867	0.8879	0.6891	0.8406	0.8440	0.8926	0.8711	0.8752	0.8740	0.8359	0.8390	0.8863	0,8861	0.8853	C288.0	10000	0.8873	0.8877	0,8874	0,8832	0.8829		0.8906	0.8912	0,8698	0 8688	0.8370	0.8370	0,8368	0.8854		0.8800	0.8883	0.8883	0.8736	0.8755	0.8742	CC88.0	0000000	0.8745	0.8746
Configu	DHI EQV	37.329	37,344	37,289	37.245	37.265	555.15 011 TI	225.275	37.231	37.218	37.220		37.069	37,354	37,393	37,369	37.337		407 - 75 402 - 72	34.739	34.730	34,579	34.630	110°45	14.643	34.656	34.667	34.671	24 630	34.576	34,567	37,290	51.512	104012	11.760	31.756	31,635	31,616	31.610	31.684	6/0°15	31.767	31,782
neters,	DH EQV	33.059 33.086	33.113	33.111	33,116	55.130	775. 22 727 22		32.432	32,573	32,532	010 0M	31.101	33,108	33,132	33.081	33,063	201.00	10.816	30.839	30,818	30.542	30.574	20 845	30.852	30,885	30.152	50.124	121.04	28.941	28,925	33,018	10.55	201 ° 00	28.214	28.208	27,636	27.679	27.634	28.056		27.780	27,795
ice Paran	T0 E0V	2180.18 2174.46	2177.85	1776.36	1784.60	1/85.50	1901.12	1980.31	2380.41	2390.83	2388.08 2520 64	2548.31	2562.20	2171.16	2169.38	2169,33	2169.90	<101,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00 20,00000 20,0000 20,00000000	2031.78	2029.50	2034,26	1629.56	1628,78	1967 91	1827.03	1826,45	2208,30	2207.84	204 022	2386.60	2386,28	2168,97	2172.28	1910 - C 1 2	1850.86	1851.85	1469,42	1473.06	1471.95	1041,08	10201	2031.49	2031,93
Performan	WAN EUV	1470.27 1474.89	1474.62	1740.66	1734.50	1/38,50	1010,90	1609.56	1329.17	1329.05	1329.20	1186.13	1186.18	1478.50	1479.92	1478.10	1477.50	6/°10†1	14/0,05	1471.83	1468.15	1737.38	1738.62	141.00	1609.82	1609.58	1330,90	1330.68	10011001	1185.00	1184.95	1476.99	1470.40	14/0,50	1469.16	1468.94	1717.24	1716.71	1715.87	1002 04	100°0701	1332.35	1332,15
lated	WA EQV	27,981	27,984	27.460	27,472	201 - 12	201.12	27.789	28.066	28,065	28,069	28,096	28.090	27,980	27.971	27,976	196.12	214.12	27.971	27.966	27,973	27,358	27,347	012.12	27.745	27.723	28.054	28,062	28,001	28.090	28,095	27,989	27,000	000 LC	27.896	27.904	27,152	27,161	27.166	605.1X	200° 12	28.048	28,041
nd Calcu	N FOV	3152.76 3163.14	3161.72	3803.39	3788.19 3707 65		34/0.1C	3475.19	2841.50	2841.34	2841.32 2677 17	2533.06	2533,68	3170.50	3174.49	3170.12	5168,19	0C.0/10	3152.52	3157.77	3149.06	3810.25	3814,56	71,1205	3481.37	3483,58	2846.44	2845.13	2512	2531,16	2530,58	3166.16	5165.cd 54 54 54 54 54	167.80	3159.90	3158,52	3794.69	3792.34	3789.79	5488.55 2488.55	01.01.00	2850.12	2850.39
Data aı	PT0/PS3	3.834 3.836	3.838	3,825	3.817	070 ° °	0.050 t	1.830	3,827	3,825	3,825	5 8 0 5	5.817	3, 839	3.846	3,842	5.856 625	200°2	101.0	3.99	3,398	3.389	3,397		3.390	3,392	3,392	292.5		3.386	3,385	3,827	120°2 -		2,990	2,990	2.993	2,991	2.990	2000	0000	2.988	2.990
ed Test	PT0/PT3	3.482 3.484	3.484	3.477	3.471		5.405 7.482	191	3.469	3.467	3.467	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	3.446	3.486	3.491	3.488	5 4H3	007°,	007 ° 0	3.143	3,142	3.124	3,130	121.5		3.133	3.134	3,135		3.123	3,122	3.477	5.480 5.62		2.804	2.803	2.790	2.788	2,788	06/ 2	2.744 3.705	2.804	2.806
Reduc	P10	29,94 29,94	29,95	29.95	29,95	27 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	20 07	29,98	29.90	29,90	29,89 20,89	29,89	29.90	29,88	29,89	29.89	29°8°	20 00 20 00	29,87	29.87	29.87	29,86	10,05		68.62	29, 9.0	29,89	29,89	20,00	29,88	29,88	29,88	40°62	20 8 8 0 C	29,88	29,88	29,89	29,89	29,89	29,05		29.89	29,89
able I.	PC1 NDES	99 100	100	120	120	021		011	06	96	06		60	100	100	100	100		66	100	66	120	120	110	011	110	00	0.0		08	80	100	001		100	100	120	120	120	011		06	96
Ľ	RDG	40 7	41	4 S	9 r 7	- 4	00	20	51	52	- - 	7 57 N 67	36	57	58	6- i	9		0 0	94	65	66	67	0 0	20	11	22	5.7	1 U	10	17	18			28	63	94	85. 9	86 96			06	16

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(ddd	PLOMANG	9.88	9.42	0 7 6			9,25	13,54	15.45	20.37	20.34	20,33	32,91		26.14	26.15	26,14	14,16	14,23	9 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		56.9	10.8	8,85	20,16	29.21	43,52	43,53	17,06	37,07	37,05	21.15	21.79	15,06	15,04	15.04			6.17	9.29	9,15	11.31
IA (PPP)	U/C0	0.3633	0.3226	0.5220		0.3700	0,3697	0.2961		0.4273	0.427.4	0.4272	0,5132	0.5111	0.4701	0.4703	0.4704	0.3848	0.3847	0 • 5 4 C 5	0.3423	0.3696	0.3706	0.5702		0.4660	0,5593	0.5588	0.5114	0.5117	0.5118	1619.0	0.4188	0.3729	0.3728	0,3728	0.3706	0012.0	0.3625	0.3624	0.3627	0.4352
ation]	ETA IT	0.8749	0,8447			0.8866	0,8861	0,8295	10000	0.8874	0,8876	0,8869	0.8504	0.8595	0.8763	0.8764	0,8766	0.8801	0.8804	0.8545	0.8541	0,8853	0.8857	0.8854	0.8740	0.8746	0.8302	0.8296	0.8546	0.8555	0,8550	1288.0	0.8828	0,8714	0.8713	0.8710	0,0000	0.8860	0.8847	0,8845	0.8851	0.8912
Configur	DHI EQV	31.776	31,687	31 . 090	100 11	37.289	37,309	37,151	27 145	28,285	28,251	28,288	26.019	28.014	28,202	28,203	28,193	28.302	28,303	28, 230	28.333	37,273	37.283	37.327	23,815	23,810	23,373	23,385	22.639	23.627	23,640	25.414	23.968	23,951	23,959	23,952	37,206	111.75	38.737	38.733	38.725	38,774
eters, (DH EQV	27,801	26,764	101.05	540.55	33,060	33.058	30,816	20.802	25,100	25,076	25,089	20 004	24,078	24.714	24.718	24.715	24.910	24.410	24.2.22	24.198	32,999	33,020	020 .22	20.815	20.825	19.404	10,401	20.202	20.213	20.211	211,12	21.158	20.870	20.876	20.862	55.042 77 070		34.271	34,260	34.271	34.555
te Param	TO EQV	2031,02	2208,46	2207 06	2170.65	2173.70	2175.06	2540.73	22.95.29	1633,66	1633,06	1633,27	10,5051	1260.98	1437.79	1436,89	1437.18	1818.75	1010.45	1993.80	1993,18	2173.17	2166.97	2170,64	1320.51	1321.68	982,84	983 27 083 21	1141.73	1142,47 .	1141.40	CA 0101	1522.25	1710.17	1710.08	1709.36	0C.7015	2167.05	2251,28	2251.57	2251,23	1858.75
rformanc	WAN EQV	1332,87	1184,00	1104.40	1476.95	1474.72	1474 .64	1185.20	1185.14	1460 87	1460.73	1460.94	104.10	1693.94	1580,49	1581.05	1581.71	1326.87	1220 10	1184.68	1184.57	1473.57	1477,12	14//.00	1427.06	1427.97	1640.78	1039.65 478 48	1532.48	1533.52	1533.75	C0 0111	1310.53	1179.54	1179.04	1178.73	C0.//#1	1478.53	1476.10	1476.05	1476.92	1740.28
lated Pe	WA EQV	28,040	28,087	28.088	27.981	27,981	27,990	28,089	28,089	27.708	27,715	27.711	24, 768	26.764	27.247	27,242	27.252	27,969	404 ac	28.066	28.068	27.992	27,977	27.051	27.027	27.051	25,905	25,903 25,803	26.444	26.455	26.446	27.583	27.592	27,936	27,926	27,925	27.080	27.974	27.981	27.987	27,986	27,493
d Calcul	N EQV	2852.10	2529.33	212722	3167.03	3162,30	3161.11	2531.64	2531.55	3163,42	3162.37	3163,20	2170,52 1708 16	3797.54	3480.30	3482,28	3482.47	2846.47	14.0482	2532 64	2532.19	3158.53	3167,85	3167.28	3166.88	3167.27	3800,35	5797,92 2706,40	3477.07	3478,05	3479.76	2840.60	2849.83	2533,35	2533.24	25-52.00	516/ 00 1170 74	3171.20	3165.23	3164.44	3166,41	16.1616
Data an	PT0/PS3	2,989	2,980	2 979	3.828	3,827	3,832	5.830 2.830	3,827	2.600	2,597	2.600	2 50H	2.598	2,600	2.600	2,599	2,595 2,595	575 C	2.596	2,595	3,825	5,826 1	1010	2.197	2,197	2,196	2,197	2.198	2.197	2,198		2.201	2,195	2.196	241 . 2	2 8 7 6	3.831	4.111	4.110	4,108	4,105
ed Test inued).	PT0/PT3	2,805	2.796	26170	3.478	5.477	3,480	3.458	3.457	2.465	2,462	2.465	2.441	2,441	2.457	2,458	2,457	2.466	0 1 1 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	2.470	2.469	3.474	3.476 7 487	2,102	2.103	2.102	2,071	<. 011 071	2.090	2.089	060°2		2,114	2,113	2.113	511°.	5.474 7 484	3.480	3.687	3,686	3.685 7 480	3.692
Reduce (Conti	PT0	29,89	29.87	29,87	29.90	29,90	29,90	06 62	29.90	29,90	29,89	06 ° 62	29.90	29.90	29,91	29,91	29,90	24 42	20 00 00	29.92	29,92	29,91	24 42	29.95	29,97	29,96	20.02	20.05	29,99	29,98	24°42	29.99	29,98	29,94	29,94	14.47 10.01	20.00	29,93	29,98	29,98	14 42	29.97
ble I.	PCT NDES	06	5		100	100	100	000	80	100	100	001	120	120	110	110	110	0 ()	80	08	80	100	001	100	100	100	120	021	110	110	011	06	06	80 	0.0	60		100	100	100		120
Та	RDG	26	м с С	5 U 6	96	47	80	6 6 6 6 6	101	102	103	104	106	107	108	109	110		711	115	116	117	2110	120	121	122	123	1 25	126	127	2 2 2 1 7 2 7	130	131	132	133	104	92	137	138	139		142

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Table I.	

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0/00	4 2 2 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
ETA 11	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
DHI EQV	38,738 38,738 38,778 38,778 38,592 38,652 38,652 38,455 38,455 38,455 38,455 37,238 37,33837 37,33837 37,33837 37,33837 37,33837 37,33837 37,3383757,
DH EQV	и и и и и и и и и и и и и и и и и и и
TG E8V	1859,66 2049,72 2054,79 2054,79 2053,56 2435,66 2435,56 2435,56 2415,55 2615,53 2615,555,555,5555,5555,5555,5555,5555,55
HAN EQV	1740,26 1616,75 1612,75 1612,75 1532,76 1332,78 1332,78 11342,78 1184,99 1184,99 1184,97 1184,781184,78 1184,78 1184,
NA EQV	228 222 222 228 228 228 228 228 228 228
N EQV	2551.02 2689.01 2485.07 2849.12 2849.99 2859.99 2551.09 2551.09 2551.09 2551.09 2551.09 2551.00 2551.00 2551.00 2163.61
PT0/PS3	444444444 •••+ •••+ •••+ ••• ••• ••• •••
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	PLT)	FLON		• • •	10.	21.	22.	22.	18.	17.	18			17.	17	32	9		10	16.	16.	18.		27.	27.	27.		21.	16.	0	0.4	16.	16.	16.		212	21.	n.		27.	27.	2	
	ddd) A	U/C0	0122 0	0.3702	0.3695	0.4448	0.4446	0,4448	0.4075	0.4076	0.4078	0.115	0.3355	0.2964	0.2964	0.3357	0.3325	1111 0	0.3694	0.3700	0.3693	0.3850	0,5044	100-4014 0-4014	0.4619	0.4618	0.4630	0.4237	0.3462	0.3464	0402.0	0.3078	0.3080	0,3698	74027	0.4021	0.4027	0.4824	0.5390	0.4431	0.4430	0.4436	
	tion 2/	ETA TT	0080 0	0.8819	0.8816	0.8880	0.8876	0.8874	0.8893	0,8895	0,8894	0.8597	0.8602	0.8243	0.8237	1.1287	0,8609	0 00 00	0.8828	0.8837	0.8830	0.8870	0,0000	0.8861	0.8858	0.8863	0.9414	0.8914	0.8674	0.8676	0.8342	0,8345	0,8340	0,8835 0,8811	10069.0	0.8894	0.8901	0,8810	0.880/ 1.0865	0.8889	0,8892	0.8076	5 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	nfigura	HI EQV		1201120	37.100	37.031	37.026	37,019	37.075	37,121	37.105	31.161	7,065	37.000	36,981	29.482	37.243	C01.16	37.128	37.151	37,132	34.520	34.514	34.346	34,363	34.379	54.300		34,560	34.586	54°267 14 482	34.504	34.489	37,145	1001	31.659	31.637	31,305	31.327	31.477	31.479	31.405	510.15
	ers, Co	H EQV D	141	2.736	2.707	2.884	2.866	2.851	2.971	3.020	3,005	870	1.882	0.500	0.460	3,277	2,061	1.4/0	2.777	2.831	2.786	0.619	000000 00000 00000	0.434	0.440	0.469	0.710	0.736	9,976	9,991	4 4 4 V	8,795	8.765	2,819		9,159	8,161	7.580	7.656	7.980	1.990	7.979	0/0./
	Paramet	G EGV DI			55.98 3	65.97 3	65.07 3	64.38 3	56.57 3	57.08 3	56.31 3	50,65 5 71,87 3		14.23 3	12.26 3	92.51 3	12.95 M		10.18 18	58.42	60.44 3	10.13 3	10.17	30.57	29.21 3	30.00	10,000	16.91	95,18 2	94.55 2	72 17 2	74.93 2	72.01 2	58,56		10 10 10 10 10 10 10 10 10 10 10 10 10 1	44.35 2	68,75 2	67,17 Z	47.31 2	47.81 2	47.51 Z	2 02 62
	rmance	EQV TO			3.23 21	9.72 170	7.71 17	9.07 170	2.31 19	2.84 19	3.62 19			52.73	5.78 25	4.33 23	2,58 23	1 45 25	10 00 S	5.48 21	2,69 210	5.04. 20	19°1		1.82 16	1.46 16		8.71 18	1.14 21	1.69 21	12 59.1 70 81 0	1.29 23	4,63 23	4.38 21 2.46		8.22	9,80 18	4.74 14	6.77 14	8.66 16	8.19 16	8.81 10	0.00 20.0
•	d Perfo	NAM VOI		107 107	02 147	173	121 123	162 173	191 561	191 291	191 161	100 11 11 12 12	221 291	93 118	01 118	504 103	71 133		147 147	97 147	147 147	147	147	172 172	173	173		10 160	133	152 133	221 120	94 118	811 58	182 147	141 01	146	146 146	171	171 171	567 159	563 159	121 154	201 101
	culated	NA E	v ac 71		59 28.0	59 27.4	17 27.4	57 27.4	48 27.7	20 27.7	79 27.7			13 28 0	80 28.1	63 Z4.5	32 28.0		0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	28 27 9	65 27.9	0.3 27.9	93 27 . 9	74. 27.3	23 27.3	10 27.3	1.12 P(28 27.7	91 28.0	73 28.0	70 28°0	21 28.0	33 28,0	14 27.9	12 21 B	80 27.8	22 27,8	94. 27.1	85 27.1	45 27.5	04 27.5	32 27.5	0°97 88
	and Cal	3 N EQ.			3156.6	3799	3798.1	3799	3480.4	3483	3483		2010	2532	2531.0	2532	2848		1512	3163	3156.6	3165.	3163.8	3793	3798	3798.7	5465. 7482	3483	2846.	2848	20400	2530	2530.6	3161.		3159.5	3163.	3788.	3793	3479.4	3479.	3479	C340.
	Data a	PT0/PS	368 1		3,821	3,819	3,819	3,818	3,816	3,823	2 8 5 1		3.819 5.819	3.822	3,818	2.713	3,852	120.0	3,825	3,830	3,826	3,391	191.5 192.2	3, 393	3.394	3,397	5,540 7,700	3,292	3,395	3.394	202.2 7.785	390	3,387	3,827	100 0	2,996	2,993	2,993	2,996	2.991	2,991	2,987	C + Y + J
	ed Test	PT0/PT3	1 161		3.450	3,441	3.440	3,439	3.447	3,453	3.451		1446	3.437	3,434	2,575	W. 470		M 10 10	3 458	3,455	3.116.	5,110	3.095	3,097	3,099	011 0 7	3,112	3,121	3,122	2117 2117	3,114	3,113	3,457	101 0	2,793	2.791	2,756	2,758	2.774	2.774	2,772	C ~ 1 ~ 2
	Reduce	014	10 00	20 02	29.92	29,87	29,89	29,87	29.87	29,88	29,88	99.90	20.89	29,91	29,90	29.09	29,91	10.00	26.92	29.92	29,91	29,93	20.00	29,94	29,94	29.94	20 00	29.94	29,93	29,93	20,00	29,92	29,92	29,95	20.00	29.95	29,94	29,95	24.95	29.91	29,92	24.42	67,70
	le II.	PCT NDES	. 00	1001	100	120	120	120	110	110	110		0	80	80	08	00	D C 7 0	100	100	100	100		120	120	120	011	110	90	00		80	80	100		001	100	120	021	110	110	110	2
	Tab	SDG	2 4		515	16	117	18	519	20		200	20	225	226	227	820	202		32	233	70	22	20	38	33		1 4 4	543	444		101	648	6 d 0		100	54	255	220	5.9	53	560	

Reduced Test Data and Calculated Performance Parameters, Configuration 2A (PPPLT) Table II.

	101)	(nangri						۳.	•			
PTO	-	PT0/PT3	PT0/P33	N FOV	WA EQV	HAN EQV	TO Eav	DH EQV	DHI EGV	(ETA TT	0 J N	FLOWAN
29,93		2.799	2,996	2528.72	28,097	1184.17	2202.21	26.672	31.717	0.8409	0.3218	14.09
29,92		2.800	2,996	2528,96	28,086	1183.82	2203,75	26.704	31,721	0,8418	0,3218	16.22
24.42		247.45		14, 4262	610.82	1105.72	2202,20	20,046	31,704	0.8419	0,3220	16.10
		201° 1	2000 1 826	00.0017	21.412	14/1.54	01.2015	200 V		0,8835	0.3691	16.46
20.04		3.457	1.827	1160.79	27.978	1011 AB		124 CT			21/240	
29.97		3.468	3,837	3165.19	27,981	1476.10	2159.10	32,867	500.75	0.8829		
29,98		3,459	3.827	3163,82	27,982	1475.49	2156.61	32.815	37.163	0.8830	0.3701	94.21
29,98		3,460	3,827	3164,54	27,986	1476.05	2157.03	32,823	37.167	0.8831	0.3702	15.47
29,98		3,467	3 638	3795.79	27,453	1736,77	1773,15	32,993	37,221	0.8864	0.4437	2.91
29,98		3.466	3 837	3795,07	27,456	1736.64	1772,99	32,979	37,213	0.8862	0.4436	19.29
29,99		3,465	3,835	3796,58	27.450	1736,95	1772,75	32,995	37,204	0,8869	0.4439	19.31
29,98		3,464	3,631	3481.33	27.790	1612,42	1964,40	33,117	37,200	0,8902	0.4071	16.64
29.97		3.461	3.827	3480.33	27.789	1611.93	1963,35	33,090	37,176	0.8901	0.4071	16.75
29,99		3,466	3.834	3481,12	27,785	1612.03	1964,07	33,115	37.214	0,8899	0.4070	16,65
29,99		2,454	2,599	3166.09	27,681	1460.66	1624.71	25,008	28,167	0,8879	0.4278	25,18
20,02		2,456	2.601	3163.79	27,691	1460.15	1627.59	25,025	28.184	0.6879	0.4273	25,24
~~~~~		0 1 2 N	100.4	01.0010	201.12	240045	1020,50	25,025	28.183	0 8879	0.4272	25.20
20,00		545°	2,000	3745.30	20.784	1693,32	1262,38	24,060	27.83	0.8645	0.5123	37,48
50,01		2,426	209 2	3794.09	26,786	1693,83	1263.11	24.077	27,843	0.8647	0,5123	37,59
10,02		2,425	2,601	3793,67	26.781	1693,29	1261.69	24,052	27,836	0.8641	0,5123	37,58
24,42			196.5		27.236	1581,29	1427.42	24,569	27.951	0.8790	0.4713	21.45
20,02			C 40 4	24/0 04 24/0 20	VC2, 12	15/9,02	1423.31	200,92	27.75	0.6740	0.4700	31,48
10.00				2877.61	27.957	1200.17	1436,00	247.45		0.0700	70/6400	10,10
29.93		2.460	2.597	2855.10	27.951	1330.12	1809.59	20.874	28.21	0.8810		
29.93		2.461	2,598	2841.97	27.951	1323.95	1813.92	24.819	28.244	0.8788	0.3840	19.83
29,93		2,462	2,595	2536,93	28,046	1185.85	1981.47	24,120	20.247	0.8539	0.410	16.65
29.93		2,462	2,596	2536,24	28,054	1185.86	1982,25	24,116	28,256	0,8535	0,3428	16.61
29.94	-	2.463	2.597	2537,35	28,050	1186,22	1981,91	24,126	28,266	0 0 0535	0.3429	16,55
29,94	-	3.467	3,837	5163,80	27,972	1474.97	2161,33	32,897	37,218	0,6839	0,3698	15.60
6 6 0 6 0		3,472	5,843	3163,72	27,975	1475.08	2163,41	32,925	37,251	0.8838	0,3696	15,58
	<b>n</b> -			C7 . C015			2105.22	32,448	195.72		0*3094	15.44
	• -	110 ° 1	001 ° 2	0/ 1010	27 011	1462,00	12.4141	100.02		100.0	1995-0	37.39
0	-	2.079		3168.69	27.000	1426.38		20.718				
29.9	-	2.096	2.187	3796.32	25.894	1638.38	981.67	19.368	101.10	0.8163		
29.93	_	2.097	2.189	3795.67	25.906	1638.86	982.75	19.377	23.742	0.8162	0.5596	14.41
29,93		2,099	2,191	3794.90	25.901	1638,20	984.70	19.415	23.764	0.0170	0.5592	
29,94		2.109	2.195	3481.93	26.440	1534.35	1137.27	20.155	23.906	0.8431	0.5125	26.01
29,94		2,109	2,195	3482.24	26,429	1533,86	1137.66	20.172	23,907	0.8438	0.5126	90.54
29,94		2,110	2,196	3481.99	26.434	1534.04	1137.41	20.163	23,923	0.8428	0.5124	79.97
29,95		2,117	2,196	2846,67	27,556	1307.48	1514.59	21.057	24.009	0.8771	0.4189	14.09
29,95		2,117	2,197	2848,05	27,571	1308.71	1514,65	21,056	24.011	0.8769	0.4190	13.92
29,96		2,086	2,197	2847,92	27,562	1308,22	1514.40	21,058	23,588	0.8927	0.4189	35,34
29,94		2,106	2,193	2530,74	27,916	1177.48	1702.45	20.770	23,861	0.8704	0,3727	20.63
29,94		2,107	2,194	2531,32	27,911	1177.52	1702.01	20.773	23,871	0.8702	0,3727	20.70
29,95		2,106	2.194	2530,89	27,911	1177.32	1702,71	20.778	23,864	0.8707	0.3726	20,01
29,94		3,458	3,825	3163,31	27.968	1474,51	2157,20	32,835	37.156	0.6837	0.3701	15,57
29,93		3.461	3,830	3166,21	27,979	1476.46	2156,58	32,842	37.179	0,6833	0.3703	15,53

Reduced Test Data and Calculated Performance Parameters, Configuration 2A (PPPPLT) (Concluded). Table II.

	FLOWANG	15.56	15.48	15.60	15 e 42	17.58	17.59	17.56	15.67	15,58	15.65	17.08	16.68	17.02	20.27	20.33	20.39	15,50	15,38	15.44	15,44	15,48	15.43	15,45	15.48	15,52	21,65	21.68	21,52
	0/0	0.3702	0 • 3 • 3 5	0.3631	0.3630	0.4343	0.4337	0,4342	0.3976	0.3978	0,3987	0.3263	0,3256	0,3261	0,2903	0.2900	0,2903	0,3699	0.3700	0,3699	0,3699	0.3700	0.3700	0,3700	0,3699	0.3700	0.4022	0,4023	0.4023
	ETA TT	0.8834	0.8817	0,8818	0,8816	0.6889	0,8884	0.8893	0.8906	0.6905	0.8910	0,8585	0.8576	0,8582	0,8210	0.8208	0.8207	0.8836	5288,0	0.8834	0,6831	0,8833	0.8840	0,8835	0.8835	0.8834	0,8897	0.8898	0.8895
	VD3 IHO	37.189	28.483	38.534	38,572	38.651	38,759	38,670	38,619	38,630	38.618	38,434	38,483	38.446	38,304	38,289	38,206	37,157	37,159	37.166	37,152	37.146	37.141	37.147	37,163	37,151	31,640	31,637	31,649
	DH EQV	32,855	33,932	33.977	34,004	34,358	34.432	34,390	34,395	34,398	34.408	32,994	33,005	32,995	31,448	31.428	31,356	32,832	32,819	32.832	32,808	32,812	32,831	32,821	32,835	32,819	28.151	28,151	28,152
	<b>TG</b> EQV	2157,36	5257.29	2230.63	2231,57	1850.35	1854,44	1852,47	2047,59	2046.74	2042,29	2417.53	2421.42	2418,45	2590,46	2592,98	2586,93	2158 64	2157.37	2158,02	2157,04	2157,89	2158,66	2158,31	2159.56	2158,22	1846.29	1846,71	1845.87
	WAN EQV	1475,85	1477.59	1476 88	1476,79	1736.81	1736,82	1737,35	1607,69	1608,70	1612.19	1330.34	1328,37	1329.87	1184,95	1183.96	1183,76	1474.24	1474 .64	1474.05	1474.01	1474.46	1474,28	1474.57	1474,68	1474.54	1468.81	1469.43	1469.06
	WA EQV	27,973	21,962	27,980	27,974	27,482	27.483	27,489	27,799	27,801	27,796	28,055	28,052	28,055	28,073	28,084	28,081	27,976	27,977	27,970	27,974	27.982	27,977	27.982	27,985	27,982	27,889	27,899	27,888
	N EQY	3165.60	3107,84	3167,01	3167.51	3791,92	3791.83	3792,11	3469.97	3471,93	3479,99	2845,17	2841,26	2844.16	2532,53	2529,45	2529,26	3161,83	3162,57	3162,10	3161,58	3161,64	3161.80	3161.88	3161,75	3161.83	3159,94	3160.24	3160,59
	PT0/PS3	3, 831	C 20 ° 7	4,096	4.103	4.110	4.132	4.114	4,105	4.107	4,105	4,093	4.102	4,095	4,099	4.097	4_080	3,825	.3,825	3,826	3,824	3,823	3,822	3,823	3,826	3,824	2,994	2,994	2,995
· /nannt	P10/P13	3.463	740.5	3,656	2005	5.674	3,690	3.677	3,669	3.671	3,669	3,641	3,649	3.643	3,622	3,620	3,608	3,458	3.459	3,460	3.458	3.457	3.456	3,457	3.459	3.458	2.791	2.791	2,792
	PT0	29.93	54.45	29,98	20,00	29,98	29,98	29,97	29,97	29,97	29,96	29,92	29,91	29,91	29,91	29,91	29,94	29,92	29,92	29,93	29,93	29,92	29,93	29.93	29,93	29,93	29,92	29,92	29,92
	PCT NDES	100	001	001	001	120	120	120	109	110	110	96	90	90	8 n	80	80	100	100	100	100	100	100	100	100	100	100	100	100
	RDG	317		212	220	321	322	323	324	325	326	327	328	329	330	331	332	333	335	336	337	338	339	340	341	342	343	344	345

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Tabl	e II	.1.	Reduc	ed Test	Data a	nd Calcu	ula ted	Performan	ice Paran	neters,	Configu	iration	3A (PPT	, (qlq
RDG	PCT P	NDES	P10	PT0/PT3	PT0/PS3	N EQV	MA EUV	MAN EUV	TG EQV	DH EQV	OHI EGV	ETA TT	07/0	FLOWANG
212	01	2	00 00	1.467	1.827	1160-97	27.918	1470 78	5161.13	CI0 CI	81C 11	A RAR	0 3408	
414			30.00	3.470	3.830	3159 64	27.919	1470.25	2163.67	32.952	37.237	0.8849	0.3696	11.42
415	10	00	30.00	3.466	3,825	3160.44	27.921	1470.72	2161.28	32.922	37.212	0.8847	0.3698	11.15
416		50	29,96	3.468	3,823	3796.21	27.297	1727.09	1767.31	33.075	37.225	0.8885	20440	17.21
417	10	50	29,97	3,471	3.827	3795,83	27,293	1726.68	1766,22	33,056	37.250	0.8874	0.4441	17.12
418	12	50	29.97	3.472	3,828	3797,38	27,304	1728,05	1766,45	33.062	37,257	0.8874	0.4442	16.99
419	11	01	29,99	3.479	3.836	3476,58	27,681	1603,93	1966.95	33,245	37.302	0,8912	0.4064	14.50
420	11	10	30.00	3.478	3,834	3475.06	27,679	1603.11	1966.55	33.226	37,295	0.8909	0.4063	14.16
421	11	10	30,01	3.476	3.831	3473.79	27,675	1602.26	1965,43	33,200	37.282	0.8905	0.4063	16-21
422	<del>ر</del> .	00	29,99	3,463	3,833	2843,50	28,047	1329,19	2357,00	32,158	37,187	0.8648	0.3325	14.63
423		ŝ	30,00	3,459	3,828	2843,34	28,041	1328,85	2354,79	32,133	37,164	0.8646	0,3326	14.59
424	U" (	0	29,99	3.461	3,830	2842,41	28.045	1328,59	2356.64	32.143	37.177	0.8646	0.3324	14.53
425		0	29,94	3 4 4 0	3,825	2527.45	28,090	1183.27	2534,56	30.690	37.024	0.8289	0,2957	17,55
426		0	29,96	5.440	3,826	2528,56	28,078	1183.27	2533.48	30.703	37.026	0.8292	0.2958	17.79
427		0	29,95	3,442	3.827	2528,56	28,081	1183.41	2534,29	30.710	37.040	0.8291	0,2958	17,52
928		0	20,00	144	5.837	3160.41	27,931	1471.24	2164.67	32,961	37,269	0.8844	0.3694	13.62
224			24.42	5.475	2.5.5.2 2.5	5100.53	27.958	1471.05	2167,32	32,995	37,275	0.8852	0.3694	13,98
	1	0.0	29,94	3.480	3.846	51.94.15	27.936	1471.16	2168.27	33,004	37,316	0.8845	0,3691	13.70
431		0	24.46	151.5	101 - 101	5101.39	27,896	1469.85	2010.22	30.749	34.644	0.8876	0.3845	14.22
	-			0.1.0	200	21,0012	12 000 FC	5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	2010102	50 154	050.95	0,8873	0.5544	14.22
		5		101.0	2 0 0 4 F	5100°/0	240,12	1404.20	50102 50 8071	10°-/54	240.05	2/00 0		14.27
	• •			611 I		1401 60	1120/2	1718 20		199 0F	24°016		000700	00° 17
	• •		00 00	111		179 547		1718 50	102010		410°30			
217		> c	50.00			1077.10	27.508	1500 04	1820.00	30,930 10 876				
438		0	29,96	3,136	3.403	3476.81	27.603	1599.50	1820.27	30.855	34.681	0.8897	0.4225	
439		2	29,96	3,136	3.403	3475.82	27.600	1598.90	1820.18	30.647	10.684	0.8894	0.4223	
440		00	29.97	3.133	3.399	2843.68	28.041	1328.99	2208.27	30.137	34.653	0.8697	0.3457	
441	•	00	29,98	3.134	3.400	2843,90	28.038	1328.94	2208.99	30.153	34.666	0.8698	0.3456	13.48
442	~	00	29.97	3.133	3,399	2843,31	28.042	1328.85	2209.20	30.145	34.659	0.8698	0.3456	14.71
443	÷	90	29,97	3,129	3,404	2526,93	28,088	1182.92	2394.14	28,986	34,625	0,8371	0.3070	15,15
444		30	29,97	3,128	3.402	2527.49	28,081	1182.90	2393,85	28.996	34,615	0.8377	0.3071	15,01
445	ب	0	29.97	3.127	3.401	2527.43	29,087	1183.13	2393,60	28,986	34.605	0.8376	0.3071	15,31
0 I 0 I 0 I		0	24,94	3.462	3,823	3159,73	27,929	1470.81	2164.78	32,958	37,186	0.8863	0,3698	14.00
				さんさっつ	1000	11.4515		14/0 01	2165.89	32.463	37.273	7788°0	0,3093	13.54
000		2		2 C Q O		1150 CE	C+4 / 7	1411.11	0C*0012	101.36	1/2°/5			
450		200	29,91	2.807	2,999	3159.61	27.823	1465-16	848.39	90.90	11.788			
451	-	0	29.91	2.806	2.998	3159.15	27.828	1465.19	1850.06	28.264	31.783	0.8893	0.4019	60.91
452	15	50	29,90	2,178	2,986	3791,24	26.987	1705.26	1457.77	27,559	31,514	0.8745	0.4831	27.47
453		50	29,90	2,784	2,993	3791.76	26,985	1705,36	1457.07	27,552	31.574	0.8726	0.4827	27.37
454	27	0	29,90	2.783	2,992	3791.29	26,993	1705.61	1456,97	27,539	31,562	0,8725	0,4828	27,36
455		0	29,95	2,801	000° M	3476.00	27,440	1589.72	1649.14	28.113	31.735	0.8859	0.4422	21,87
828 81		0	29,95	2.801	2,000	5476,25	27.459	11.9851	1650,37	28,137	31.738	6988.0	0.4422	21.91
457	-		24,44	202.2	5005 C	5476,23	27,443	66°69CI	1051,22	28.147	151.15	0.5505	0.4420	21.90
	~ 0	0.0	14.45	2,000	777.N		774°17	01.0241	74°,7202	21,000	200-15	10-0-0	<105°0	
	- 0		07°70		1 4 4 4 V	2041-07 1201		C/ 02CI	24.0402					
40C			0 1 0 C	100.0	044	2641.46	24,076	1142.45	00 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		51.17U	0 0 0 0 C	5 1 C C C C	10.41
292			26,95	2,805	2,998	2527.03	28.072	1182.33	2222.47	26.923	11, 775	0.8473	0.3215	
1		2		) > ) <b>a</b> a		1 2 <b>-</b> 1 2 1 2	•••	>>====		1 J J J		1 - 1 > + >	トーようテン	

Tab	le III.	Redu (Con	ced Test tinued).	. Data a	nd Calcı	lated ]	Performa	nce Para	meters,	Configu	ration	3A (PPT	PLP)
RDG	PCT NDES	PT0	PT0/PT3	PT0/PS3	N EQV	NA EQV	WAN EQV	TO EQV	DH EQV	DHI EQV	ETA 11	070	FLOWANG
463	80	29,95	2.804	2,997	2527.09	28.072	1182.35	2222.09	26.920	31.766	0.8474	0.3216	11.99
464	100	29,96	3,476	3,838	3157,09	27,928	1469.54	2169.62	33,005	37,281	0,8853	0.3690	13.44
465	100	29,96	3,473	3,835	3157,37	27,928	1469.63	2168.70	32,995	37,264	0.8854	0.3691	13.64
100	001	44.45 10.00	5.477	0 * 0 * 0 0 * 0 * 0	3157,66	27,925	1469.62	2169,68	33.016	37,293	0,8853	0.3690	13.42
101		50 0C		2 5 5 5 4	00,1010		24.0241	1025,18	<10°52	28.196	0.8872	0.4269	22,60
	001	40 00	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	5000 5000	02./clc	100°''		1023.40	+ 10°C>	20.211	10000	0.4268	22.65
470	120	29,96	2 4 35	100	70. COTA	10.10	CA 0041	C2 7201	54°44	C/1°07	C/00°0		00127
471	120	29.96	2.435	2.596	3792.51	26.545	777.87	1001001	000.10	010 70	0.8562	0.5126	
472	120	29,96	2.435	2.597	3792.27	26.563	1678.93	1245.53	23.929	27.953	0.8561	0.5125	14.45
473	110	29,96	2,446	2,593	3479.02	27,073	1569.79	1421.76	24,587	28,075	0.8758	0.4705	28.03
474	110	29,96	2.446	2,593	3480,17	27,069	1570.09	1421.05	24,586	28,073	0.8758	0.4707	28,01
475	110	29,96	2,446	2,593	3479,29	27.076	1570.09	1421.36	24.579	28,072	0.8756	0.4706	28,83
01	0.0	24,45	2.462	2,594	2847,94	27,932	1325,81	1816.95	24.930	28,255	0.8823	0,3851	17,35
1/4		c, , , , , , , , , , , , , , , , , , ,	20405	292 <b>.</b> 2	2849.03	27,928	1326.14	1816.27	24.934	28,261	0.8823	0.3852	17,36
180		10.00	2 440	102 0	10,4445	200 042 200 042	1104.50	99°9661	201 - 104	242.02		0.5424	15,08
100	001		1 1 1 1	140.2	0244C2	200.02	14.0011	1444.16	24°240	242.85	1959°0	0.5425	15.04
483	100	29,91	3,466		20 9912			C104,035	760°55	143 <b>6</b> 75	0 0000	20/5*0	
484	100	29.93	3,453	3.809	3165.02	27.925	1473.05	2156.08		211212	0.88.0		
485	100	29,93	2,090	2.186	3165.19	26.806	1414.08	1300.58	20.667	23.639	0.8743	0.4670	
486	100	29,92	2.093	2,190	3166.79	26,824	1415.76	1303.80	20.714	23,683	0.8747	n.4668	31.65
487	100	29.93	2.096	2,193	3167.12	26,834	1416.43	1305,50	20.736	23.722	0,8741	0.4664	31,62
489	120	29,93	2,068	2,191	3796.03	25,619	1620.87	964.54	19.233	23,333	0,8243	0,5593	43,89
500	1 4 0	24.45	2,067	191.5	3796.12	25,616	1620.67	965,35	19.252	23,326	0,8253	0.5594	43.88
141		27°23	2 000		3462.10	26.175	1519.05	1122,26	20.091	23,483	0.8556	0.5138	38,01
202				1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	12,2040	101.02	07.40 101	00.1211	110.02	7/C. 52	CIC9.0	92140	38.00
101		10.02	0110	2,200	28.404	27.444	11.63011	C2.1211	<0.040 040 040 040	1/0.62		1510.0	58,01 27,14
495	6.6	29,94	2.110	2,199	2846.87	27.447	1302.30	1515.05	21.147	C10 . SC	0.8844	0.4186	
496	96	29,94	2.111	2.200	2846.78	27.451	1302.47	1515.84	21.155	23,929	0.8841	0 4184	24.30
497	80	29,93	2,111	2.196	2532,37	27,863	1175.97	1708.76	20.900	23,929	0.8734	0.3727	18,25
498	0	59.95	2.111	2,196	2532,04	27,868	1176.07	1710.13	20.910	23,929	0.8738	0.3726	19,21
1 4 4	001	20 07	211.2	2.140 1 835	00.[202	200 12	69°0/11	1109.67	20.908	22.454	0,8728	0.5724	18,17
	100	16.02	1.470		10,0010	016 12 21 016	14/6.10	900,301, 36 1,40		5/•C10	0,00,0	C0/C*0	12.1
502	100	29.82	3.075	3.517	3163.97	27.486	1449.42	2173.60	33.674	14.174	0.9854		
503	100	29,94	3.628	4.044	3163.19	27.886	1470.16	2221.21	33.907	19.344	0.8843	0.3640	11.85
504	100	29,94	3,616	4.030	3164 00	27,923	1472.45	2226,35	33,950	38.264	0.8872	0.3645	14.08
505	100	29,79	3.227	3,542	3164.60	27,315	1440.70	2236.63	34,871	35.414	0,9847	0.3794	26,06
506	120	29,92	3.672	4.088	3798,83	27.331	1730.42	1839.45	34.407	38,643	0.8904	0.4358	15,01
507	120	29,93	3.673	4,089	3798.19	27,331	1730.16	1840.07	34.412	38.646	0.8904	0.4357	15,00
200		24.42	5,0//	4°04'	5798.60	27.327	1730.08	1841.65	34.451	38.674	0.8908	0.4356	15,10
	011	10 00	0.0.0		24/4°30	100.17	1002.44	2030.00	54.480	30,028	0,8920	1645.0	13,43
		10.00	0.0°0	1000	000 1000 7471 4545	100.12	1000	20.00.05 77 8700	14.440	50°059	0.553.0	2005 0	15.42
512	06	29.93	3,645	4 . 086	2847.80	28.029	1370 36	2127.70		18 457	0.8631	7445.0	
513	9 0 0 6	29.93	3,653	4,099	2847.53	28.031	02 0221	2428.69		18.545	1770°0	0.3264	
514	Ü6	29,94	3.649	4 0 0 6	2846.90	28.066	1331.70	2426.10	33.117	38.489	0, 8604	0.3264	15.62
515	80	29,92	3,622	4,093	2530.41	28,072	1183,91	2604.08	31,588	38,306	0,6246	0.2902	19,56

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(dla)	FLOWANG	19.76 19.61 13.45 13.26			· • •				
3A (pp ⁻	03/0	0.2911 0.2903 0.3696 0.3696 0.3697 0.3701							
ration	ETA IT	0,8262 0,8250 0,8854 0,8856 0,8856	· .						
Configu	OHI EGV	38.264 38.260 37.262 37.235							
meters,	DH EQV	31,615 31,566 32,991 32,976 32,893							
lce Para	TO EQV	2600,03 2603,47 2164,52 2164,10 2158,19 2158,19			•		. ·		·
Performar	WAN EGV	1187.24 1184.07 1470.67 1470.30 1470.23							
ula ted ·I	WA EUV	28.078 28,081 27.912 27.912 27.909				•			
nd Calc	N EQV	2536,99 2529,99 3161,32 3160,53 3160,61	• •	·					· · ·
Data a	PT0/PS3	4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0			· ·			•.	
ced Test luded).	£19/019	3.616 3.616 3.473 3.469 3.469	•						
Reduc (Conc	01d	29,92 29,92 29,95 29,95 29,95							
le III.	PCT NDES	80 80 100 100							-
Tab	. 90	20							

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A (PPTI	0.00	0,3693 0,3696	0.3704	0.4070	0.4078	0.3336	7555 0	0,2961	0.2961	004290	0.3700	0.3695	0,3698	0, 3698	0.3851	0,3855	0.3848	107°0	10110	0.4233	0.4232	0.4235	0.3457	0.3079	0,3082	0.3713	0.3711	0.3712	0.3634	0.3629	0.4353	0.4352	0.3997	0.3997	1997.0	2726.0	0.3272	0.3700	0.3703	0.3698
ation 4	ETA IT	0.8849 0.8849	0.8855	0,8451	0.8926	0.8664	0 8444	0.8305	0.8306	0.8862	0.8456	0,8855	0,8858	0.8858	0.8885	0,8893	0.8883	0.8855	0.8829	0.8917	0.8915	0.8916	0.8712	0.8399	0,8402	0.6874	0.8875	0.8876	0.8860	0.8855	0.8912	0.8902	0.8933	0,8929	0.8931	0.8545	0.8676	0.8866	0,8862	0.8860
Configur	OHI EQV	37.300 37.305	37.287	125.75	37,317	37,222	202°/2	37,112	37,090	51,107	37,316	37,341	37,332	565.75 562.75	34 717	34,680	34.745	545 45	34.563	34.654	34.675	34.683	34.723	34.619	34.615	37.222	37.232	37.271 38.581	38.762	38.787	38.774	38,802	38,728	38.702	38.701	56.553	50,516 38,576	37.280	37.271	37.281
eters, (	DH EGV	33,006 33,010	33.017	15.4.65	33,307	32,248	56.510 107 cr	30,823	30,808	50°746	33,049	33.065	33.070	100°55	30,644	30 841	30,865	000000	30.517	30.902	30,913	30.425	30.249	29.076	29.084	33.031	33,044	33,083 TA TAK	34,342	34.344	34 554	34,543	34 597	34.557	34,565	33.512	33.200	33.053	33.028	33,030
e Parame	TO EQV	2171.93 2169.71	2166,42	1965_40	1966.00	2358,22	2142 10	2542,49	2542.02	2168.51	2168.57	2171.94	2171.08	2171.30	2018.77	2017.98	2020.56	42°4201	1628.12	1821.60	1822.51	1621.64	2215.39	2396,84	2395 <b>,</b> 63	2160.61	2161.89	2162.73	2245.18	2247,46	1849.01	1848.05	2042.16	2041.75	2042.31	2435.07	2434.01	2169.93	2167.97	2169.75
erformanc	WAN EQU	1471.27 1471.71	1474.61	1609.86	1608,85	1332.69	10,0001	1184.68	1183.97	1472.83	1473.41	1471,81	1472,44	1473.38	1472.49	1473.05	1471.85	10, 2011	1720.32	1601.13	1601.54	1002.04	1328.76	1183.98	1185,14	1474.34	1473.78	1474.71	1476.31	1475.17	1732.49	1732.70	1609.01	1609.25	1608.90	c/ 2551	10,5221	1472.19	1473.69	1471.14
ated Pe	WA EGV	27,960 27,948	27,951	27.686	27,691	28,052	28.040	28,090	28,086	200,02	27.940	27,940	27.938	27,938	27.896	27.897	27.893	212°12	27.223	27,606	27.611	21.611 28 020	28.031	28.072	28.075	27.905	27,903	27,900	27.916	27,919	27,359	27.359	27.692	27.708	27.705	28.047	28.051	27.935	27,947	27.934
d Calcul	N EQV	3157,28	3165,42	3488,86	3486.03	2850.51	750.0502	2530,48	2529 <b>.</b> 32	3164.31	3164.11	3160,66	3162,23	3164.25	3167,14	3168.17	3166.10	C. 24/5	3791.64	3479,47	3480.17	5463.07	2844.14	2530,57	2532,80	3170.07	3169.13	3171,42	3173.03	3170.27	3749.39	3749.98	3486.18	3484.77	3484,31	11,1285	2849,95	3162.01	3163, A5	3159,91
Data an	PT0/PS3	3,828 3,828	3,825 2,023	3,826	3,827	3,824	0.00°.	3,823	3,819	1.829	3,831	3,835	3,833 * 870	3.839 839	3 395	3,390	665 N	122.5	3.388	3,393	3,396	5,597 7 707	398	3,390	2,589 2,882 2,882	5.812	3,813	3,820	4.112	4,118	4,105	4,110	4.094	4,089	4,089	4.005	1001	3,824	3.822	3,823
d Test	PT0/PT3	3.478 3.479	3.476	1910 1980	3,481	3.467	. 477	3,452	3.449	3.480	3,481	3.484	3,483 7 487	3.487	3.140	3,136	3.144	20100	3.122	3.133	3.135	071.7	3.141	3.128	5,128	3.467	3,469	3.663	3.690	3.694	3,692	3.696	3.685	3.681	3.681		3,665	3.475	3.474	3.476
Reduce	P.T.O	29,93 29,93	29,93	29,90	29,90	29,92	29,92	10, 91	29,91	29,91	29,90	29,90	06 62	29,90	29,91	29.91	29,92	20.00	29.92	29,94	29,94	20.01	29,93	29,95	20.00	29,99	29,98	29,94	29,95	29,95	20,00	29,96	29,91	29.90	29,91	00°00	29.90	29,89	29.90	29,89
le IV.	PCT NDES	100	100	110	110	00	06	80	0 0 0 4	100	100	100		100	100	100	001	120	120	110	110	011	0.6	80	000	100	100	001	100	100	021	120	110	110	110		0.6	100	100	100
Tab	RDG	154	158	163	164	168	170	171	172	140	175	176	1/1	179	180	181	281	180	185	186	187	001	190	192	194	195	196	197	199	200	201	202	204	205	505		209	210	211	212

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Table V.	

FLOWANG	NUNO 010 000 000 000 000 000 000 000 000 00
0/00	0,2961 0,2961 0,2962 0,2962 0,2962 0,29663 0,2962 0,2993 0,2993 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2599 0,2596 0,2599 0,2599 0,2599 0,2599 0,2599 0,2590 0,2590 0,2590 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,2962 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200 0,200000000
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Table VI. Overall and Stage Performance Summary.

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OVERALL PER	Configuration	Work, E/9	Pressure Ratio	Efficiency, n _{mm}
-	FORMANCE			¥7
-	3-PP	13.76	1.604	0.875
1 + 2	2-PPPP	26.38	2.66	0.868
1 + 2	4-PPTP	26.78	2.66	0.880
1 + 2 + 3	l- ppppp	33.00	3.47	0.886
1+2+3	5- pppppT	32.97	3.47	0.885
1.+2+3	6-PPTPTT	32.90	3.47	0.883
1 + 2 + 3	7-pppplp	33.00	3.47	0.886
1 + 2 + 3	2A-PPPPLT	32.90	3.47	0.883
1+2+3	<b>3A-PPTPLP</b>	33.00	3.47	0.886
1+2+3	4A- PPTPPP	33.05	3.47	0.887
STAGE PERFO	RMANCE			
1	PP/	13.76	1.604	0.875
2	/44/	12.62	1.658	0.846
5	/TP/	13.02	1.658	0.873
e	/PP ¹	6.62	1.305	0.923
m	TPT /	6.62	1.305	0.918
ო	EL	6.12	1.305	0.856
3	ALP ²	6, 62	1.305	0.923
m	ΛT	6.52	1.305	606.0
3	/pp ³	6.35	1.305	0.877
1 As tested	in Configuration	1 (PPPPP).		
a As tested	in Configuration	1 (PPPPLP).		
³ As tested	in Configuration	1 4A (PPTPPP).		



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TIP





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Numbers Shown on Velocity Diagrams are Angles in Degrees and Mach Numbers

Figure 1. Turbine Design Velocity Diagrams.



Figure 2. Three-Stage Turbine Flowpath.



Figure 3. Two-Stage Turbine Flowpath.



Figure 4. Plain Blade Airfoils.



Figure 5. Stage Two Tandem Stator Assembled.



Figure 6. Stage Three Tangentially Leaned Stator Airfoils Viewed Aft Looking Forward.



Figure 7. Stage Three Rotor Plain Blade.

Figure 8. Stage Three Rotor Tandem Blade.



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Figure 10. Typical General Electric, Evendale, Air Turbine Test Facility Configuration.



Figure 11. Air Turbine Test Instrumentation.



Figure 11. Air Turbine Test Instrumentation (Concluded).









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Total-to-Total Efficiency Vs. Total-to-Total Pressure Ratio at Design Equivalent Speed. Base Cases Compared. Figure 14.



Figure 15. Equivalent Torque Vs. Total-to-Total Pressure Ratio, Configuration 1A (PPPPPP).



Equivalent Weight Flow Vs. Total-to-Total Pressure Ratio, Configuration 1A (PPPPP). Figure 16.



Figure 17. Equivalent Specific Work Vs. Total-to-Total Pressure Ratio, Configuration 1A (PPPPPP).











Equivalent Specific Work Vs. Weight Flow - Speed Parameter, Configuration 1A (PPPPP). Figure 20.



Figure 21. Equivalent Torque Vs. Total-to-Total Pressure Ratio, Configuration 2A (PPPPLT).







Figure 23. Equivalent Specific Work Vs. Total-to-Total Pressure Ratio, Configuration 2A (PPPPLT).
















Figure 28. Equivalent Weight Flow Vs. Total-to-Total Pressure Ratio, Configuration 3A (PPTPLP).



Figure 29. Equivalent Specific Work Vs. Total-to-Total Pressure Ratio, Configuration 3A (PPTPLP).







Configuration 3A (PPTPLP)









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Figure 33. Equivalent Torque Vs. Total-to-Total Pressure Ratio, Configuration 4A (PPTPPP).



Figure 34. Equivalent Weight Flow Vs. Total-to-Total Pressure Ratio, Configuration 4A (PPTPPP).



Figure 35. Equivalent Specific Work Vs. Total-to-Total Pressure Ratio, Configuration 4A (PPTPPP).



Figure 36. Total-to-Total Efficiency Vs. Blade-Jet Speed Ratio, Configuration 4A (PPTPPP).



Figure 37. Total-to-Total Efficiency vs. Total-to-Total Pressure Ratio, Configuration 4A (PPTPPP).







Figure 39. Equivalent Torque Vs. Total-to-Total Pressure Ratio, Configuration 5A (PPTP).



Figure 40. Equivalent Weight Flow Vs. Total-to-Total Pressure Ratio, Configuration 5A (PPTP).



Figure 41. Equivalent Specific Work Vs. Total-to-Total Pressure Ratio, Configuration 5A (PPTP).









Figure 44.

. Predicted and Actual Equivalent Torque Vs. Total-to-Total Pressure Ratio, Configuration 1A (PPPPPP).





















Figure 51. Normalized Static Pressure Vs. Axial Station, Configuration 1A (PPPPPP), at Design Equivalent Speed.



Figure 52. Normalized Static Pressure Vs. Axial Station, Configuration 2A (PPPPLT), at Design Equivalent Speed.



Figure 53. Normalized Static Pressure Vs. Axial Station, Configuration 3A (PPTPLP), at Design Equivalent Speed.



Figure 54. Normalized Static Pressure Vs. Axial Station, Configuration 4A (PPTPPP), at Design Equivalent Speed.



Figure 55. Turbine Efficiency Contour Plot, Configuration 1A (PPPPPP).





Turbine Efficiency Contour Plot, Configuration 2A (PPPPLT).



Figure 57. Turbine Efficiency Contour Plot, Configuration 3A (PPTPLP).







Figure 59. Turbine Exit Swirl Contour Plot, Configuration 1A (PPPPPP).










Figure 62. Turbine Exit Swirl Contour Plot, Configuration 4A (PPTPPP).



Figure 63. Turbine Exit Swirl Contour Plot, Configuration 5A (PPTP).



Figure 64. Turbine Exit Swirl Contour Plot, Configuration 2 (PPPP).



Percent Turbine Exit Height

105



Figure 66. Stage Three Incidence Angle Contour Plot, Typical for Two-Stage Tandem Turbine (PPTP).



Figure 67. Stage Three Incidence Angle Contour Plot, Typical for Two-Stage Plain Turbine (PPPP).



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