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# LONG-TERM CF6 ENGINE PERFORMANCE DETERIORATION - EVALUATION OF ENGINE S/N 451-479

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16. Abstract		3		
This report summarizes the pserial number 451-479 at the This United Airlines engine had N3029U. The investigative terefurbishment. The performs of the low pressure turbine ai the stage one fan blades. The measurements and airfoil sur cluded in this report is a detaical teardown report with a deanalytical assessment of the pconditions to losses in both SI	General Electric and completed its in st program was connectesting includer foils, and a final analytical teardof face finish checks filed analysis of the stailed descriptions	Aviation Service Op- nitial installation on onducted inbound pr- led an inbound test, test after leading e- wn consisted of deta of the as received the test cell performa- tion all observed has (deterioration) relation	eration, Ontario a DC-10 aircraft ior to normal ov a test following edge rework and ailed disassemble deteriorated har ance data, a con- rdware distress, ting measured har	o, California. number verhaul/ cleaning cleaning dy inspection rdware. In- nplete analyt- and an an ardware
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#### SYMBOLS AND ACRONYMS

A<sub>4</sub> Stage One High Pressure Turbine Nozzle Area

ALF Aft Looking Forward

ASE Airline Support Engineering

ASO Aviation Service Operation

ASO/O Aviation Service Operation/Ontario

B30 Forward Compressor Discharge Pressure Seal Clearance - reference

Shop Manual, 72-00-00

B70 Aft Compressor Discharge Pressure Seal Clearance - reference Shop

Manual, 72-00-00

B71 High Pressure Compressor Balance Piston Seal Clearance -

reference Shop Manual, 72-00-00

B/P Blue Print

BW Blade Width

C27 Low Pressure Turbine Pressure Balance Seal Clearance - reference

Shop Manual, 72-00-00

CDP Compressor Discharge Pressure

CR Cruise

CRF Compressor Rear Frame

CW Clockwise

DACO Douglas Aircraft Company

DELT Delta

DETAC Delta High Pressure Compressor Efficiency

DETALP Delta Low Pressure System Efficiency

DETALPS Delta Low Pressure System Efficiency

DFN1 Delta Net Thrust at Constant Fan Speed

Digital I Instrument that Displays in Digits

DPARA Delta Parasitics

DPARAS Delta Parasitics

E12, E13 Fan Blade Tip Clearances - reference Shop Manual, 72-20-01

EGT Exhaust Gas Temperature

EGT Margin Exhaust Gas Temperature Margin

EPR Engine Pressure Ratio

ETAC High Pressure Compressor Efficiency

ETALPS Low Pressure System Efficiency

ETAT High Pressure Turbine Efficiency

F<sub>N</sub> Net Thrust

 $F_N@N1$  Net Thrust at Constant Fan Speed

FBW Full Blade Width

FIR Full Indicated Runout

FWD Forward

G1, G2, G3, High Pressure Turbine Rotor Forward Shaft, Forward Seal Teeth

G4, G5, G6

Gilmore Tradename for Cell Meter Displaying Exhaust Gas Temperature

H1, H2, H3, High Pressure Turbine Rotor Forward Shaft - Aft Seal Teeth

H4, H5, H6

HP High Pressure

HPC High Pressure Compressor

HPS High Pressure System
HPT High Pressure Turbine

HPTN High Pressure Turbine Nozzle

HPTR High Pressure Turbine Rotor

HRS Hours

ID Inside Diameter

IGV · Inlet Guide Vane "

IN Inch

Dim "K" Dimension "K", High Pressure Turbine Nozzle Support - reference

Shop Manual, 72-52-00

LE, L/E Leading Edge

LP Low Pressure

LPS Low Pressure System

LPT Low Pressure Turbine

LPTN Low Pressure Turbine Nozzle

LPTR Low Pressure Turbine Rotor

LPTS Low Pressure Turbine Stator

MAX Maximum

M/C Maximum Continuous

MM Maintenance Manual

MRL Maximum Repairable Limit

N1, N<sub>1</sub> Fan Speed

NO. Number

No. 4B Number Four Ball Bearing

OGV Outlet Guide Van

P3, P3 Compressor Discharge Pressure

P<sub>4Q</sub>, P49 Low Pressure Turbine INlet Pressure

PARAS Parasitics

QEC Quick Engine Connect

RAD Radius

RMS Root Mean Square

SFC Specific Fuel Consumption

SFC Margin Specific Fuel Consumption Margin

SL Sea Level

S/M Shop Manual

S/N Serial Number

STG Stage

SWECO Vibratory Mill Cleaning Process

 $T_{3}$  Compressor Discharge Total Temperature

 $T_5X$  Calculated Exhaust Gas Temperature

T/C Thermocouple

TE Trailing Edge

TMF Turbine Mid Frame

TSN Time Since New

T/O Takeoff

UA United Airlines

V1, V2, V3, High Pressure Turbine Rotor Thermal Shield Seal Teeth

V4

WC16 Sixteenth Stage Cooling Air Flow

WF Fuel Flow

Δ Delta

η Efficiency

ηc High Pressure Compressor Efficiency

nf Fan Efficiency

nt High Pressure Turbine Efficiency

n2t Low Pressure Turbine Efficiency

#### 1.0 INTRODUCTION

CF6-6D Engine S/N 451-479 was selected to be the first of a planned four Task IV (Long Term Performance Deterioration) engines, in accordance with the requirements outlined in the NASA-Lewis CF6 Jet Engine Diagnostics Program, Contract No. NAS3-20631. Log No. Dll, dated March 11, 1977, describes the rationale and justification for selection of this engine. Included in that document, is a performance history of 451-479 including production acceptance test, aircraft trim run, first takeoff and cruise, and revenue service cruise trend data.

The test plan for this engine was defined in Log No. D12, dated March 11, 1977. Included in that document is a list of test objectives, a description of the basic CF6-6 engine, an itemized test plan schedule, detailed instructions for the designated performance tests, analytical teardown, refurbishment and reassembly instructions, and an instrumentation and facilities description for the General Electric Aviation Service Operation (ASO) Ontario, California CF6 Test Cell.

Log No. D13 describes the instrumentation required for the performance testing of the engine. Standard airline instrumentation was required to measure test cell engine performance. Additional low pressure turbine inlet pressure probes (P49) and high pressure compressor discharge temperatures (T3 rake) were requested and were used to ensure data consistency and accuracy.

This report summarizes all of the pertinent data generated during the course of the test plan, together with an analytical evaluation of the data. This report is a revision of the first 451-479 Task IV Engine Report dated July 29, 1977. Sections 5.0 (Performance Summary) and 7.0 (Analytical Assessment of Performance Losses) have been rewritten to reflect recent findings with regard to cruise to test cell performance correlations and errors in the analytical teardown hardware assessment.

#### 2.0 OBJECTIVES

In accordance with the requirements outlined in the NASA-Lewis CF6 Jet Engine Diagnostics Program, Contract No. NAS3-20631, the following objectives were considered paramount for engine S/N 451-479, as well as for all future Task IV engines:

- Component analyses of long-term performance deterioration with regard to deterioration magnitude and apportionment to individual components.
  - High Pressure Compressor Efficiency
  - High Pressure Turbine Efficiency
  - Parasitics
  - Low Pressure (LP) System Efficiency
  - Thrust at Fan Speed
- Evaluation of LP turbine (LPT) performance restoration with regard to LPT vane and blade surface finish.
- Evaluation of fan performance restoration with regard to blade leading edge quality and airfoil surface cleanliness.
- Analysis of HP core losses (HP compressor, HP turbine, and parasitics) for use in correlating analytical teardown inspection results.
- Obtain data for the CF6 deterioration model in terms of both component and overall (EGT and SFC) performance.

#### 3.0 ENGINE HISTORY

CF6-6D Engine S/N 451-479 was installed new on United Airlines DC-10-10 Aircraft No. N3029U (DACO #207) in the left wing location (Position No. 1). Subsequently, it was delivered to United Airlines in July 1975, with the engine still installed in its original position.

The United Airlines DC-10 route structure includes Chicago as a focal point, with routes, including intermediate stops, to the West Coast (Los Angeles, San Francisco, Portland, and Seattle) and Hawaii, and to the East Coast (Boston, New York, Washington, and Miami). The UA fleet includes 37 DC-10-10 aircraft.

Engine 451-479 was a CF6-6D "Task Force" engine (451-467 and up), which contained a number of performance and ruggedization improvements. The primary modifications included elliptical grinding of the HPT shrouds, shimming of the Stage 1 HPT nozzle, and increased cooling holes in the Stage 2 HPT nozzle support.

The engine was removed from the aircraft on March 16, 1977 to participate in the NASA-Lewis Diagnostics Program as the first Task IV engine. At that time, it had accumulated 4468 total hours and 1910 cycles. After removal of the QEC kit at UA, the engine was delivered to the General Electric Aircraft Service Operation (ASO), Ontario, California, on March 18.

# 4.0 SUMMARY OF EVENTS

Work Order No. 181460 was generated by ASO/O to fulfill the requirements of the Test Plan (Log No. Dl2). The program objectives were met in the following chronological sequence:

Date	Event
3-18-77	Engine arrived at Ontario. Incoming inspections performed and engine prepared for testing. Suspected faulty EGT T/C harness (lower left) replaced.
3-21-77	Engine set up in test cell. Initiated inbound test.
3-22-77	Completed inbound test. Engine returned to shop.
3-24-77	Initiated LPT disassembly and inspection checks.
4-1-77	Completed inspections and SWECO-cleaning of LPT blades and vanes.
4-15-77	Completed rebuild of engine.
419-77	Engine set up in test cell. Initiated retest, with cleaned LPT airfoils.
4-20-77	Completed test. Recontour and clean Stage 1 fan blades.
4-21-77	Completed test. Install original lower left EGT T/C harness.
4-21-77	Completed test. Engine returned to shop.
4-25-77	Initiated analytical teardown of core.
5-9-77	Completed core engine inspections.
5-17-77	Engine (in modules) returned to United Airlines.

#### 5.0 451-479 PERFORMANCE SUMMARY

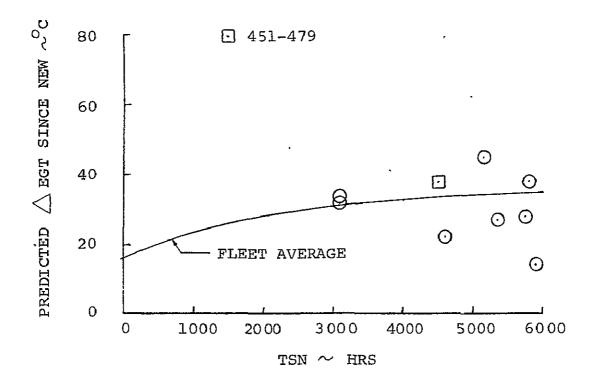
Performance testing of 451-479 consisted of three separate double-power calibrations. The three were an inbound test, a test following cleaning of the LPT blades and vanes, and final test following leading edge rework and cleaning of the Stage I fan blades. The data were consistent and repeatable with the exception of EGT which shifted 10° C cold between the inbound and post LPT performance tests.

The inbound test included two down-power calibrations. A grounded section (lower left) of the EGT harness was replaced prior to running the test. The calculated minus indicated EGT ( $\Delta T_5 X$ ) level (5° F) was consistent with the outbound Evendale production power calibration (9° F) which indicates changing this segment had a negligible effect on the measured EGT.

The measured performance deterioration (production new to ASO/O inbound) was consistent with the level obtained for eight other inbound tests. Figure 5-1 presents aircraft cruise trend data as compared to these sample engines. Shown on these curves is a "fleet average" level which corresponds to the average cruise deterioration level of CF6-6D engines S/N 451-406 and up. Figures 5-2 and 5-3 present the inbound test cell deterioration levels for 451-479 and the other eight inbound engines. The "sea level fleet model" shown is based on sea level-to-altitude correlations of 0.7 for SFC, 0.8 for EGT, and 1.0 for WF (i.e.,  $\Delta$ EGT at cruise = 0.7 x  $\Delta$ EGT at sea level).

The inbound data are summarized in Tables 5-1 and 5-2. Almost all the measured component deterioration (inbound versus production acceptance) occurred in the HP turbine as presented below. Note that 1.3% of the 4.5% SFC deterioration is due to the "instant loss" at the aircraft manufacturer and not deterioration during revenue service.

-0.2%	$\Delta$ ETAC	(HP Compressor Efficiency)
-4.3%	$\Delta$ ETAT	(HP Turbine Efficiency)
0	ΔPARAS	(Core Engine Internal Leakages and Cooling Flows)
-0.1%	$\Delta$ ETALPS	(LP System Efficiency)
+0.6%	FN@Nl	(Thrust at Fan Speed)



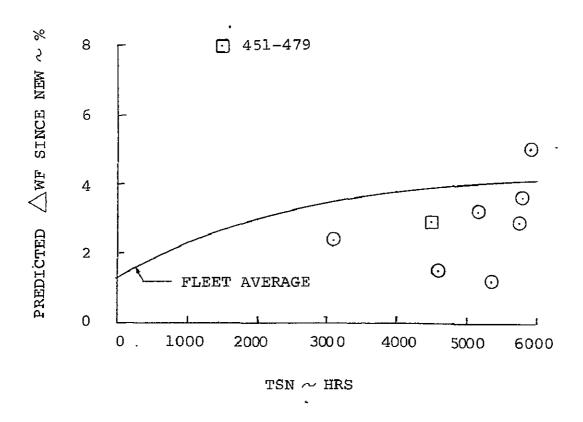
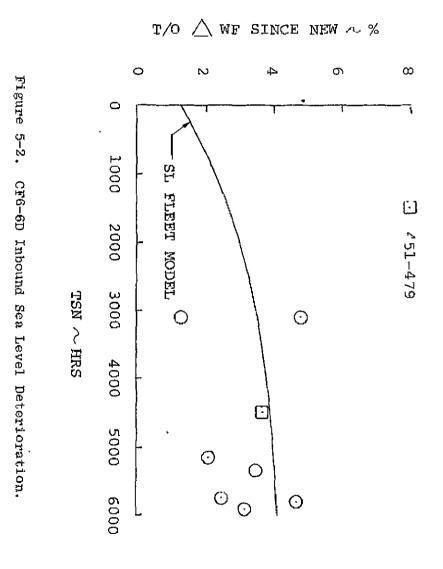
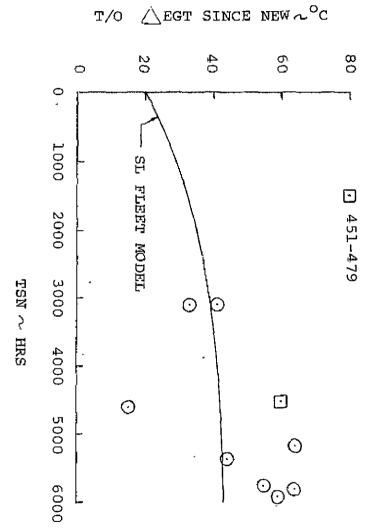


Figure 5-1. CF6-6D Cruise Deterioration.





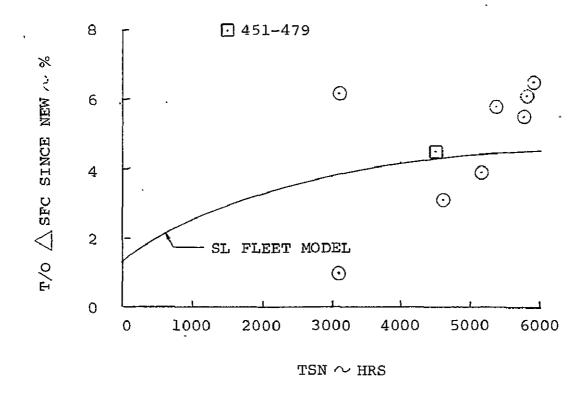


Figure 5-3. CF6-6D Inbound Sea Level Deterioration.

Table 5-1. 451-479 Average Takeoff and Maximum Conditions.

	Production	Inbound	After LPT	After Fan
Date	12-11-74	3-22-77	4-19-77.	4-21-77
SFC Margin	1.0%	-3.5%	-3.7%	-3.2%
T/O Hot Day EGT	1566° F	1674° F	1656° F	1662° F
M/C Hot Day EGT	1503° F	1606° F`	1590° F	1594° F
T/O Hot Day T5X	1557° F	1668° F	1668° F	1670° F
M/C Hot Day T <sub>5</sub> X	1493° F	1602° F	1603° F	1603° F
% DETAC	-0.1	-0.3	-0.2	-0.4
% DETAT	-1.1	-5.4	-5.5	-5.3
$\% \  ext{DPARA} \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \$	-1.3	-1.4	-1.4	-1.5
% DETALP	0.0	-0.1	-0.5	0.2
% dfnl	-0.7	-0.1	0.0	0.5
% DETAC	-0.1	-0.3	-0.2	-0.4
% DETAT	-1.0	-5.0	-4.3	-4.3
% DPARA Based on EGT .	-1.4	-1.4	-1.3	-1.4
% DETALP	0.1	0.1	0.3	0.9
% dfni	-0.7	-0.1	0.0	0.6

Note: Stackup versus an average 1975 production engine. Inbound data includes "instant" loss at the aircraft manufacturer (1.3% SFC).

Table 5-2. 451-479 Stackup.

						Based or	n T <sub>5</sub> X				Based	i on EGT		
		SFC	Hot Day				J							
Run	Reading	Margin	EGT	delt <sub>5</sub> x	DETAC	DETAT	DPARAS	DETALPS	DFN1	DETAC	DETAT	DPARAS	DETALPS	DFN1
Production	9	0.9	1566	-9	-0.3	-0.8	-1.2	-0.1	-0.6	-0.3	-0.7	~1.2	-0.0	-0.6
$A_4 = 52.561$	10	1.0	1566	-9	0,2	-1,5	-1.5	0,1	-0.5	0.2	-1.3	-1.6	0.2	-0.5
	11	$\underline{1.1}$	<u>1503</u>	<u>-10</u>	<u>-0.1</u>	<u>-1.1</u>	<u>-1.3</u>	<u>-0.1</u>	-0.9	<u>-0.1</u>	<u>-1.0</u>	-1.4	<u>-0.0</u> .	<u>-0.9</u>
	AVG	1.0	1566/ 1503	~9	-0,1	-1.1	-1.3	-0,0	-0.7	-0.1	-1.0	-1.4	0.1	-0.7
Inbound	9 .	-3.6	1672	-7	-0.3	-5.4	-1.6	-0.5	0.0	-0.3	-5.2	-1.7	-0.3	0.0
$A_4 = 53.549$	10	-3.4	1672	1	-0.6	~5.5	-1.5	0.3	0.3	-0.6	-4.8	~1.5	0.7	0.3
	11	-3.2	1606	6	-0.4	-4.8	-1.1	0.0	0.1	-0.4	-4.6	-1.1	0.2	0.1
	12	-3.7	1604	0	-0.1	-6.0	-1.8	-0.1	-0.2	-0.1	-5.4	~1.8	0.3	-0.2
	19	-3.3	1675	-8	-0.4	-5.4	-1.4	0.2	-0.2	-0.4	-5.2	-1.4	0.3	-0.2
	20	-3.6	1675	-7	-0.4	-5.1	-1.2	-0.4	-0.2	-0.4	-4.9	-1.2	-0.2	-0.2
	<u>21</u>	<u>-3.5</u>	1606	<u>-6</u>	<u>-0.1</u>	<u>-5.5</u>	<u>-1.3</u>	-0.4	<u>-0.5</u>	-0.1	-5.2	<u>-1.3</u>	-0.2	-0.5
	AVG	<b>-3.</b> 5	1674/ 1606	-5	-0.3	-5.4	-1.4	-0.1	-0.1	-0.3	-5.0	-1.4	. 0.1	-0.1

+108°	F	EGT@N1	(EGT at Far	n Spe	ed)		
+111°	F	T5X@Nl	(Calculated	i EGT	at	Fan	Speed)
+4.5%		SFC@FN	(SFC at Th:	rust)			

The measured performance stackup is presented using an A<sub>4</sub> of 53.594 sq. in. An attempt was made to measure A<sub>4</sub> but nine of the Stage 1 HPT vane segments were badly burned at the trailing edge. In addition, ballooning, cracking, and bowing of the nozzle vanes raised questions as to whether A<sub>4</sub> was assessed correctly. The resulting A<sub>4</sub> was obtained by extrapolating the measurable vane segments using tool 2C6846. A<sub>4</sub>, in conjunction with the measured HPT pressure ratio, is used to assess the performance tradeoff between parasitics and HPT efficiency. Furthermore, the condition of the TMF liner can affect the pressure (P49) as well as the temperature (EGT) measurement. Similarly, much effort has been placed in developing a simple, yet accurate, method of measuring compressor discharge temperature and pressure. A change in either parameter will result in a different component assessment as to the correct apportionment to compressor efficiency, turbine efficiency, and parasitics.

The test following the LPT airfoil cleaning also consisted of two down-power calibrations. The data indicated no improvement relative to the inbound run. In fact, the LP system efficiency worsened after the LPT work. The data analysis is slightly clouded due to a shift in indicated EGT. T<sub>5</sub>X (calculated EGT), however, was consistent and was adequate for the component stackups. The performance data are summarized in Tables 5-1 and 5-3. The following component deltas were measured relative to the inbound run:

+0.1%	$\Delta$ ETAC
-0.1%	$\Delta$ ETAT
0	ΔPARAS
-0.4%	ΔETALPS
+0.1%	FN@Nl
-18° F	EGT@Nl
0	т <sub>5</sub> х@nі
+0.2%	SFC@FN

Table 5-3. 451-479 Stackup.

				Based on T <sub>5</sub> X						Based on EGT				
		SFC	Hot Day											
Run	Reading	Margin	EGT	DELT <sub>5</sub> X	DETAC	DETAT	DPARAS	DETALPS	DFN1	DETAC	DETAT	DPARAS	DETALPS.	DFNI
After LPT	11	-3.6	1656	10	-0.1	-5.3	-1.4	-0.4	0.5	-0.1	-4.2	-1.2	0.3	0.5
A <sub>4</sub> = 53.549	12	-3.4	1655	10	-0.2	-5.2	-1.4	-0.2	0.3	-0.2	-4.1	-1.2	0.4	0.3
	13	-3.6	1590	14	0.0	-5.6	-1.4	-0.3	0.2	0.0	-4.3	-1.2	0.5	0.2
	14.	-3.5	1590	15	-0.2	~5.6	-1.4	-0.1	0.1	-0.2	-4.3	-1.2	0.7	0.1
	21	-3.7	1657	12	-0.1	-6.3	-2.1	-0.4	-0.0	-0.1	-5.1	-1.9	0.4	-0.0
	22	-3.9	1656	15	-0.5	-4.8	-0.8	-0.6	-0.2	-0.5	-3.5	-0.7	0.2	-0.2
	23	-3.7	1591	7	-0.1	-5.2	-1.3	-1.0	-0.3	-0.1	-4.2	-1.2	-0.4	-0.3
	24	<u>-3.8</u>	<u>1590</u>	<u>15</u>	0.0	<u>-6.1</u>	<u>-1.7</u>	<u>-0.6</u>	-0.4	0.0	<u>-4.8</u>	<u>-1.6</u>	0.2	-0.4
	AVG	-3.7	1656/ 1590	1.2	-0.2	-5.5	-1.4	-0.5	0.0	-0.2	-4.3	-1.3	+0.3	0.0

The double-power calibrations following the fan cleaning and leading edge rework showed a positive improvement in both SFC and thrust (0.5% SFC@FN and +0.5% FN@N1). There was no improvement in EGT@N1 because both airflow and fan efficiency increased. The performance data are summarized in Tables 5-1 and 5-4. The following component deltas were measured relative to the previous test:

-0.2%	$\Delta$ ETAC
+0.2%	$\Delta ETAT$
+0.1%	ΔPARAS
+0.7%	ΔETALPS
+0.5%	FN@Nl
+6° F	EGT@N1
+2° F	${f T_5}$ X@Nl
-0.5%	SFC@FN

Following the fan test, a complete investigation was conducted to understand the indicated EGT shift. Table 5-5 summarizes the checks made to the harness and cell readout system. The checks revealed no discrepancies. The grounded section of EGT harness was reinstalled and check points were run to try and understand the EGT shift. There was no measurable difference in EGT or  $\Delta T_5 X$  (Table 5-6). An inspection following the last testing sequence showed one of the harness probe aspirator holes to be immersed in the cooler TMF liner. Due to the mechanical condition of the liner (cracks and two holes), buckling of the liner may have caused the indicated EGT shift.

Table 5-4: 451-479 Stackup.

Based on T<sub>5</sub>X Based on EGT

		SFC	Hot Day				)							
Run	Reading	Margin	EGT	delt <sub>5</sub> x	DETAC	DETAT	DPARAS	DETALPS	DFN1	DETAC	DETAT	DPARAS	DETALPS	DFN1
After Fan	31	-3.4	1662	7	-0.6	-4.8	-1.0	-0.1	0.3	-0.6	-3.8	-0.9	0.5	0.3
$A_4 = 53.549$	32	-3.5	1658	2	-0.3	-4.9	-1.2	-0.6	-0.1	-0.3	-4.2	-1.2	-0.2	-0.1
	<b>3</b> 3	-3.4	1591	10	-0.3	-5.6	-1.7	-0.2	-0.1	-0.3	-4.5	-1.5	0.5	-0.1
	34	-2.7	1594	6	-0.1	-5.6	-1.6	0.6	0.5	-0.1	-4.7	-1.5	1.2	0.5
	39	<b>~3.1</b>	1663	6	-0.4	-5.9	-2.0	0.6	0.7	-0.4	~5.0	-1.9	1.2	0.7
	40	-3.0	1663	6	-0.5	-5.6	-1.9	0.6	0.8	-0.5	-4.8	-1.8	1.2	0.8
	41	-2.9	1594	8	-0.4	-4.9	-1.1	0.6	0.7	-0.4	-4.0	-1.0	1.2	0.7
	42	-3.5	1594	3	-0.5	-4.6	-1.1	-0.7	0.1	-0.5	-3.9	-1.1	-0.2	0.1
	43	-3.8	Ī659	16	-0.6	-5.2	-1.3	-0.2	0.7	-0.6	-3.8	-1.1	0.7	0.7
	44	~3.5	1665	16	-1.1	-5.1	-1.2	0.7	0.9	-1.1	-3.7	-1.1	1.6	0.9
	45	-2.7	1596	8	-0.1	-5.8	-1.8	0.9	1.1	-0.1	-4.8	-1.7	1.5	1.1
	<u>46</u>	<u>-3.4</u>	1594	<u>17</u>	<u>-0.2</u>	-6.1	-1.8	0.3	0.6	<u>-0.2</u>	<u>-4.7</u>	-1.6	<u>1.1</u>	1.6
	AVG	-3.2	1662/ 1594	9	-0.4	-5.3	-1.5	0.2	0.5	-0.4	-4.3	-1.4	0.9	0.6

#### Table 5-5. EGT Harness Checks.

- 1. Verify digital and cell meter (log sheets); indicate same EGT.
- la. Check same on previous run (if data are available).

#### After Next Run

- 2. Send calibration signal through cell system (at forward lead connection and read-out on Digital + Gilmore.
- 3. Visual EGT harness and leads for broken/damaged leads or connectors. Check that connectors are seated.
- 4. Check total EGT System (at forward lead connector) with Meggar per 77-21-01.
- 5. Disconnect forward and aft lead and disconnect T/C harness from aft lead.
  - Visual connectors for broken, missing, bent, or loose pins.
  - Check pin-pin and pin-casing per MM for
    - T/C harness (four sectors)
    - Forward lead
    - Aft lead

Check shop to see if anyone noted any discrepancy during disassembly and reassembly.

Table 5-6. 451-479 Stackup.

					Based on T <sub>5</sub> X					Based on EGT				
		SFC	Hot Day				-	1						
Run	Reading	Margin	EGT	delt <sub>5</sub> x	DETAC	DETAT	DPARAS	DETALPS	DFN1	DETAC	DETAT	<b>DPARAS</b>	DETALPS	DFN1
Clean Harness	50	-3.4	1669	-3	-0.3	-5,1	-1.5	-0.2	0.9	-0.3	-4.6	-1.5	0.1	0.9
A <sub>4</sub> = 53.549	<b>'51</b>	-3.3	1594	10	0.1	-5.8	-1.6	0.1	0.5	0.1	-4.7	-1.5	0.8	0.5
	<u>52</u>	<u>-3.8</u>	<u>1596</u>	<u>13</u>	-0.4	<u>-4.6</u>	<u>-0.8</u>	<u>-0.6</u>	0.7	<u>-0.4</u>	<u>-3.4</u>	<u>-0.6</u>	<u>0.1</u>	0.7
	AVĢ	-3.5	1669/ 1595	· 7	-0.2	-5.2	-1.3	-0.2	0.7	-0.2	-4.2	-1.2	0,3	0.7
01d Harness	54	-3.7	1666	4	-0.5	-5.1	-1.4	-0.5	0.7	-0.5	-4.3	-1.4	0.0	0.7
A <sub>4</sub> = 53.549	55	-3.4	1598	0	0.1	-5.4	-1.7	-0.3	0.5	-0.1	-4.8	-1.7	0.1	0.5
•	<u>56</u>	<u>-3.3</u>	<u>1596</u>	<u>7</u>	<u>-0.1</u>	<u>-5.4</u>	<u>-1.5</u>	<u>-0.1</u>	0.8	<u>-0.1</u>	<u>-4.5</u>	<u>-1.5</u>	0.4	0.8
	AVG	-3.5	1666/ 1597	4	-0.2	-5.3	<b>-1.</b> 5	-0.3	0.7	-0.2	-4.5	-1.5	0.2	0.7

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#### 6.0 POSTTEST TEARDOWN RESULTS

Upon completion of the test cell runs, the core engine was subjected to an analytical teardown inspection, the results of which are contained herein. During the course of the test program, the low pressure system had likewise been disassembled, cleaned, and inspected. These results are also included in this section.

The inspection results include observations concerning the hardware which were performance related, and do not imply that no other discrepancies existed.

#### 6.1 HIGH PRESSURE COMPRESSOR SECTION

#### 6.1.1 HP Compressor Rotor Assembly

#### 6.1.1.1 General Inspection

Inspection of the HP compressor rotor revealed the spool and blades to be extremely dirty throughout, with a heavy deposit of oil mixed with the dirt in the forward end (through Stage 6). Some very mild aluminum splatter was noted on blades in Stages 12 and aft. A photograph of the rotor assembly was taken (Figure 8-1); however, it was not taken until after the rotor had been prematurely cleaned, erasing all evidence of the dirty condition.

#### 6.1.1.2 Rotor Land Rubs

No vane-to-spool rubs were detected on any land. There was a new gouge, probably caused by a variable stator vane near the split line being out of position during disassembly.

#### 6.1.1.3 Rotor Land Coating Condition

The coating of the compressor rotor lands was inspected with the following discrepancies noted (Figure 8-2 in Appendix B):

Stage	Conditions
15 and 16	100% missing
14	20/30% missing
13	30/40% missing
3 - 12	ОК

#### 6.1.1.4 Blade Airfoil Condition

Except for the dirt and oil previously noted, the airfoils were in excellent condition. There was no evidence of any nicked, or otherwise damaged blades. Blades in Stages 8 through 12, 15, and 16 showed some slight tip rub, as evidenced by the rubs noted on the compressor stator lands (see 6.1.2.4).

#### 6.1.1.5 Blade Surface Finish

Two blades each from Stages 3 through 16 were removed for surface-finish checks, using a profilometer. Original plans also included blades in Stages 1 and 2 for these checks. However, to expedite the test plan, blades in these two stages were visually inspected and deemed to be about the same quality as blades in Stages 3 and 4; therefore, they were not removed from the rotor.

Surface-finish checks were taken at 15%, 50%, and 85% of blade height at:

- 10/15% of chord from leading edge on the suction side.
- 10/15% of chord from trailing edge on the pressure side.

Results were as follows (RMS  $\mu$  inch):

Convex —				<del></del>	Concave —					Stage	
					Stg.					Stg.	Overall
Stage	Tip	Pitch	Root	Avg.	Avg.	Tip	Pitch	Root	Avg.	Avg.	Avg.
3	25	15	13	18	•	46	30	15	30		
	21	19	21	20	19	50	26	15	30	30	25
4	29	11	14	18	_	55	26	20	34		
•	20	15	15	17	17	55	35	20	37	35	26
5	30	16	16	21		48	38	18	35		
	30	24	18	24	22	46	44	25	38	37	30
6	25	13	12	17		50	36	65	50		
	26	17	21	21	19	44	46	45	45	47	33
7	22	15	26	21		44	62	60	55		
	25	15	25	22	21	42	30	50	41	48	35
8	27	18	20	22		40	50	65	52		
	30	20	20	23	23	40	40	59	46	48	36
9	40	30	23	31		39	35	35	36		
	34	34	29	32	32	39	42	50	44	40	36
10	30	30	35	32		40	42	50	44		
	27	35	·40	34	33	37	40	45	41	42·	38
11	30	30	26	29		30	38	45	38		
	35	30	20	28	29	37	40	25	34	36	32
1.2	30	30	32	31		35	41	45	40		
	30	25	27	27	29	40	38	55	44	42	36

Convex Stg.							← Concave ← →					
Stage	Tip	Pitch	Root	Avg.	Avg.	Tip	Pitch	Root	Avg.	Stg. Avg.	Overall Avg.	
13	30	23	22	25		35	37	40	37			
	35	26	25	29	27	45	40	38	41 '	39	33	
14	30	32	24	29		37	40	50	42			
	27	23	20	23	26	30	40	33	34	38	32 `	
15	34	40	27	34		40	37	40	39			
	41	30	25	32	33	35	30	45	37	38	36	
16	27	28	24	26		45	33	40	39			
	31.	60	60	50	38	32	22	32	29	34	36	

Average blade surface finish = 33 RMS  $\mu$  inch.

#### 6.1.1.6 Rotating CDP Seal

Measurements of each of the CDP seal teeth were made, with the results as follows:

Tooth	Max. Diameter	FIR
I	18.130"	0.004"
Н	17.931"	0.002"
G	17.731"	0.0015"
F	17.531"	0.001"
E	17.3305"	0.001"
D	17.131"	0.0015"

Resultant clearances can be found in 6.1.3.3, "Stationary CDP Seal Measurements."

#### 6.1.2 High Pressure Compressor Stator Assembly

#### 6.1.2.1 General Inspection

Inspection of the HP compressor stator assembly revealed a heavy mixture of oil and dirt in the forward stages, through Stage 6. The heaviest buildup accumulated on the IGV's and the IGV inner shrouds, and got progressively better from stage to stage. The remaining vanes, though not oily, were extremely dirty. There was a very mild splattering of aluminum on the Stage 13 vanes and OGV's. A photograph of the upper stator case can be seen in Appendix B, Figure 8-3.

#### 6.1.2.2 Condition of Variable Stator Bushings

Variable stator bushings were generally in excellent condition. A total of 12 Stage 6 vanes were found to be slightly loose and only one vane in this stage revealed metal-to-metal contact. Stage 5 had three vanes which were slightly loose, but no metal-to-metal contact.

All of the Stage 3 vanes, in the lower case only, were on the loose side. This would seem to indicate that they had been initially assembled in this manner, since all other variable vanes in both casing halves exhibited the normal tightness.

#### 6.1.2.3 Vane Airfoil Condition

Except for the oil and dirt, all airfoils were in excellent condition. No nicks or other discrepancies were noted on any vane.

#### 6.1.2.4 Casing Rubs

Inspection of the stator case lands revealed the following blade tip-to-case rubs:

HP Compressor Casing Rubs - Upper (See Photo, Figure 8-4 in Appendix B)

<u>Stage</u>	Depth	<u>Width</u>	<u>Location</u>	Remarks
1-7	No Rub			
8	Kiss	FBW	12 o'clock	3" in length no depth
	.003	${ t FBW}$	1 o'clock	1.3" in length
9	Kiss	FBW	9-10 o'clock	1.5" in length no depth
10	No rub			
11	Kiss	•5 <sup>11</sup>	1 o'clock	2.5" in length no depth
12	Kiss	• 411	12 & 1 o'clock	
				only. No depth
13	No rub			
14	No rub		•	
<b>15</b> .	.005	FBW	10-11 o'clock	4.5" in length
	.003	FBW	12 o'clock	2.3" in length
	.005	FBW	1-2 o'clock	4.5" in length
16	008	FBW	1-2 o'clock	4.5" in length

Note: FBW = Full Blade Width

HP Compressor Casing Rubs - Lower (See Photo, Figure 8-5 in Appendix B)

Stage	<u>Depth</u>	Width	Location	Remarks
1-7	No rub			
8	Kiss	.050"	7-8 o'clock	6" in length, T/E
9	No rub			
10	Kiss	1/2 BW	7 o'clock	1.5" in length
11	No rub			
12	No rub			
13	No rub			
14	No rub			
1.5	Kiss .	1/2 BW	5-7 o'clock	12" in length
	.003	FBW	8 o'clock	2.5" in length
16	.003	FBW	5-6 o'clock	3" in length

Note: FBW = Full Blade Width

#### 6.1.2.5 Vane Surface Finish

Two vanes each from Stages 7 through OGV's were removed for surface-finish checks. Readings were taken with a profilometer at 15%, 50%, and 85% of vane height at:

- 10/15% of chord from LE on convex side.
- 10/15% of chord from TE on concave side.

Results are as follows (RMS-  $\mu$  inch):

	<del></del>	<del></del> -	Convex		<del></del>	<del></del>	<del></del> -c	oncave		<del></del>	Stage
					Stage					Stage	Overall
Stage	Tip	Pitch	Root	Avg.	Avg.	Tip	Pitch	Root	Avg.	Avg.	Avg.
7	25	40	40	35		130	<b>7</b> 0 -	65	88		
	35	35	35	35	35	140	65	70	92	90	63
8	65	60	55	60		75	60	45	60		
	70	75	55	68	64	85	70	65	73	67	65
9	40	40	35	38		<b>7</b> 5	45	55	58		
	55	40	45	47	43	65	55	50	57	58	50
10	40	45	40	42		60	50	50	53		
	40	35	40	38	40	50	50	50	50	52	46
11	30	35	.30	32		40	50	50	47		
	30	35	30	32	32	55	40	40	45	46	39
12	35	40	35	37		60	50	60	57		
	25	25	35	28	33	50	40	55	48	53	43
13	35	35	40	37		45	55	55	52		
	35	35	45	38	38	50	45	55	50	51	44

← Convex ← → → → → → ← ← ← ← ← ← ← ← ← ← ← ← ←				← Concave ← → →				Stage			
Stage	Tip	Pitch	Root	Avg.	Stage Avg.	Tip	Pitch	Root	Avg.	Stage Avg.	Overall Avg.
14	25 35	30 40	35 35	30 37	33	50 50	40 45	60 55	50 50	50	42
15	35 35	35 35	35 35	35 35	35	45 50	50 40	45 40	47 43	45	40
OGV	40 35	40 40	40 45	40 40	40	35 40	45 35	35 45	38 40	<b>3</b> 9	40

Average surface finish of vanes = 47 RMS  $\mu$  inch.

#### 6.1.3 Compressor Rear Frame

#### 6.1.3.1 General

A cursory inspection of the compressor rear frame revealed no notable discrepancies. A more detailed inspection of the combustor and dimensional checks of the CDP seal and the 4B pressure balance seal (mini-nozzle) were performed, with results as follows:

#### 6.1.3.2 Combustor

To expedite the work scope, the combustor was not removed from the rear frame, but rather visually inspected as installed. The combustor had its typical 1/4" to 1/2" cracks, inner and outer liners. The most noteworthy fault was the cracks originating at the inner liner's aft thimble louver at approximately the 7 o'clock location. This particular louver had cracked in four different directions of varying lengths (Figures 8-6 and 8-7 in Appendix B). The cracks were approximately 5/8", 1", and 2-1/2" in length and another about 2-1/4" long connecting with a 4-1/2" circumferential crack. The whole piece was bulged into the airstream approximately 5/8". There were six more of these thimble louvers with cracks ranging from 1/2" to 1-1/4" but these are considered insignificant with regard to performance.

#### 6.1.3.3 Stationary CDP Seal, Forward

Measurements of each of the lands on the 4R CDP seal were taken and recorded. (A view of the forward CRF seals can be seen in Appendix B, Figure 8-8.) The results were as follows (all dimensions are in inches):

Dia.	F1	F2	F3	F4	<b>F</b> 5	F6
Max.	18.149	17.950	17.750	17.550	17.352	17.152
Min.	18.148	17.949	17.749	17.549	17.351 ·	17.151
3	18:148	17.949	17.750	17.549	17.352 <sup>-</sup>	17.152
4	18.149	17.950	17.749	17.550	17.352	17.152
5	18.149	17.950	17.750	17.550	17.352 ·	17.151
6	18.148	17.950	17.749	17.549	17.351	17.152
Avg.	18.148	17.949	17.749	15.549	17.352	17.152

#### Shop Manual Dimensional Requirements

Maximum	18.152	17.952	17.752	17.552	17.352	17.152
Minimum	18.148	17.948	17.748	17.548	17.348	17.148

Clearances determined from this data and that taken on the corresponding HPC rotor seal (6.1.1.6) were as follows:

	<u>B30</u>							
	1	2	3	4	5	6		
Minimum	0.007	0.008	0.0085	0.0085	0.0095	0.0095		
Maximum	0.0115	0.0105	0.010	0.010	0.011	0.011		
Average	0.009	0.009	0.009	0.009	0.010	0.010		

Overall average clearance = 0.0093

#### 6.1.3.4 No. 4B Pressure Balance Seal Measurements

Diameter measurements of each land of each of the aft seals in the No. 4B pressure balance seal (mini-nozzle) were taken and are recorded below (dimensions are in inches). These are the seals that mate with the high pressure turbine rotor forward shaft seals. (A photograph of the mini-nozzle is contained in Appendix B, Figure 8-9.)

# Forward Seal (Aft CDP Seal)

			<u> 1</u>	<u> 870</u>	•	
Dia.	F1	F2	F3	F4	F5	<b>F</b> 6
Max. Min. 3 4 5 6 7 8 Avg.	7.944 7.942 7.942 7.942 7.942 7.943 7.943 7.944 (7.943)	8.104 8.102 8.103 8.103 8.104 8.104 8.103 8.102 8.103	8.265 8.263 8.263 8.265 8.265 8.264 8.264 8.265 8.264	8.424 8.422 8.424 8.423 8.424 8.423 8.422 8.422	8.584 8.582 8.582 8.583 8.582 8.584 8.582 8.583	8.744 8.742 8.743 8.742 8.744 8.743 8.743 8.743
	<u>S1</u>	nop Manua	al Dimens	sional Re	equiremen	its
Maximu	m 7.945	8.105	8.265	8.425	8,585	8.745
Minimu	m 7.942	8.102	8.262	8.422	8.582	8.742

### Aft Seal (HPT Balance Piston Seal)

<u>B-71</u>							
Dia.	F1	F2	F3	F4	<b>F</b> 5	F6	
Max. Min. 3 4 5 6 7 8 Avg.	10.443 10.442 10.443 10.443 10.443 10.443 10.443 10.425	10.622 10.616 10.616 10.617 10.621 10.622 10.620 10.620 10.619	10.778 10.770 10.772 10.771 10.773 10.770 10.770 10.778 10.773	10.942 10.939 10.940 10.940 10.941 10.941 10.939 10.942	11.105 11.098 11.100 11.098 11.099 11.101 11.104 11.105 11.101	11.260 11.249 11.255 11.258 11.260 11.258 11.262 11.262	
Shop Manual Dimensional Requirements							
Maximum	n 10.440	5 10.606	5 10.76	6 10.92	6 11.08	6 11.246	
Minimun	n <b>10.</b> 442	2 10.602	2 10.76	2 10.92	2 11.08	2 11.242	

Resultant clearances between these seals and the corresponding rotating seals are contained in Section 6.2.3.6, "Forward Shaft Seal Dimensions."

#### 6.2 HIGH PRESSURE TURBINE SECTION

#### 6.2.1 Stage 1 High Pressure Turbine Nozzle Assembly

#### 6.2.1.1 General Inspection

Visual inspection of the Stage 1 HPTN assembly revealed 13 vanes with some degree of burning on the trailing edge, while some 10 other vanes were bowed in the same area. (See photographs in Appendix B, Figures 8-10 through 8-13. Note that the position markings as viewed in the photographs are incorrect. Per B/P, the vane marked No. 18 is really Vane No. 1. From that point, the vanes should be marked in clockwise order. For orientation purposes, all references to vane position numbers in this report are per the actual position and not as depicted in the pictures.) As can be seen in the photographs, all vane distress emanated approximately from the center of the airfoils at the trailing edge, and continued radially outward and inward about the same amount for each particular vane.

All of the vanes that experienced the burning/distortion showed radial cracks of varying lengths on the concave side, depending on the severity of the distress. These cracks were located 3/8" to 1/2" forward of the trailing edge.

The leading edges of the vanes exhibited the normal minute cracks and splatter buildup, but all cooling holes appeared to be open. The same could be said for the concave face of the vanes; while the convex side was smooth, which is also normal. Surface finish checks of the airfoils were made on several vanes and are tabulated in 6.2.1.5. "Airfoil Surface Finish Checks."

The thermal shields, inside the vane platform ID, were distorted the full 360° circumference. Two bolt head covers, No. 2 (in line with Nozzles No. 5 and 6) and No. 12 (in line with Nozzles No. 45 and 46), were also heavily deformed.

The aft face of the Stage I vane outer hook showed contact 360°; however, the first vane of each segment (CW, ALF) appeared to be marked heavier than the other.

The inspection also revealed (though it was not performance-related) that

five of the outer fishmouth seal tabs were burned away (three approximately 25% missing; one approximately 50% missing; one approximately 10% missing). There was also one inner fishmouth seal tab burned with approximately 40% missing.

# 6.2.1.2 Drop Dimension - CRF to Stage 1 Vanes

Drop dimensions from the CRF aft flange to the aft face of the Stage 1 vane outer hook were taken at 16 equally spaced locations, starting at 12 o'clock and working CW. The results were as follows:

1	4.868"	9	4.853"
2	4.860"	10	4.868"
3	4.864"	11	4.859"
4	4.850"	12	4.869"
5	4.858"	13	4.869"
6	4.856"	14	4.869"
7	4.853"	15	4.867"
8	4.856"	16	4.860"

Average = 4.861"

Note: This engine was equipped with the 0.020" shim (PPN 92570), which mounts between the CRF/Stage 2 nozzle flanges. Stackup of mating parts is contained in 6.2.2.5.

## 6.2.1.3 Vane Outer Platform Gap Measurements

Gaps between outer platforms on adjacent vanes were measured at 16 equally spaced locations and were as follows:

1	0.029"	9	0.029"
2	0.025"	10	0.029"
3	0.025"	11	0.032"
4	0.027"	12	0.030"
5	0.027"	13	0.032"
6	0.034"	. 14	0.035"
7	0.025"	15	0.027"
8	0.025**	16	0.027"

These are well within the shop manual limits of 0.015" min. and 0.045" max.

## 6.2.1.4 Area Check (A4)

A check of the nozzle exit area was conducted with some difficulty. Due to the distress previously noted, it was impossible to obtain an accurate reading on some of the severely burned and distorted vanes. Measurements were taken of all the vanes that it was possible to measure, and data extrapolated to arrive at the final estimated A<sub>4</sub> of 53.960 sq. in.

Area check of each of the individual nozzles is as follows:

Nozzle		Nozzle		Nozzle		Nozzle	
No.	Area	No.	Area	No.	Area	No.	Area
1	D	17	0.822	33	0.835	49	0.835
2	0.885	18	0.820	34	0.841	50	0.825
3	0.833	19	0.828	35	0.827	51	0.845
4	0.865	20	0.799	36	0.808	52	0.793
5	0.880	21	0.824	37	0.856	53	0.823
6	D	22	0.834	38	0.891	54	0.838
7	D	23	0.808	39	0.930	55	0.825
8	0.882	24	0.798	40	0.854	56	0.789
9	0.857	25	0.832	41	0.917	57	0.820
10	0.835	26	0.870	42	D	58	0.845
11	0.888	27	0.837	43	$\mathbf{D}_{\cdot}$	5 <del>9</del>	0.832
12	0.814	28	0.784	44	D	60	0.804
13	0.838	29	0.836	45	D	61	0.821
14	0.824	30	0.848	46.	0.868	62	0.852
15	0.838	31	0.832	47	0.870	63	D
16	0.785	32	0.796	48	0.832	64	D

D = Distorted and/or Burned

## 6.2.1.5 Airfoil Surface Finish Checks

Three nozzle segments were removed from the assembly to check the airfoil surface finish. No problems were encountered in measuring the finish of the convex side; however, due to the curvature of the vane, it was not possible to set up the equipment to obtain all the desired measurements on the concave side. Measurements were taken at the pitchline at 10%, 50%, and 90% chord.

The following are the results of the surface finish measurements (RMS):

	<del></del>	Conv	/ex	<del></del>	← Concave →			
Vane No.	Fwd.	Mid	Aft	Avg.	Fwd.	Mid	Aft	Avg.
1	45	65	60	57	170		250	210
	35	55	65	52				
2	50 ·	40	35	42	180		280	230
	45	25	35	35				
3	35	35	65	45	170		500	335
	35	40	35	37		<u></u>		
	Overal	1 Avera	age	44				258

## 6.2.2 Stage 2 High Pressure Turbine Nozzle Assembly

#### 6.2.2.1 General Inspection

A visual inspection of the Stage 2 high pressure turbine nozzle, as assembled, produced no surprising results. It proved to be about as expected for an assembly with this amount of running time (see photograph, Figure 8-14, for an overall view of the assembly). The more noteworthy results of the inspections are covered in the following paragraphs.

## 6.2.2.2 Shroud Rubs and Condition of Bradelloy

The Stage 1 shrouds were very rough and eroded/oxidized. One shroud (No. 17) had an irregular shaped piece missing, approximately  $1" \times 1-1/2"$  (see photograph in Appendix B, Figure 8-15). This was the only missing section.

Additional roughness was noted on the shrouds due to burning, located axially in the center of several of the pieces. Burned areas averaged approximately 3/8" x 1-1/2" on each of 13 shrouds with only three parts showing any depth (~0.010"/0.020").

The Stage 2 shrouds had light erosion/oxidation, with no missing pieces. All shrouds were heavily rubbed, with a more recent rub, approximately 1"  $\times$  1-1/2", noted on shroud No. 1 (see photograph, Figure 8-16, in Appendix B).

#### 6.2.2.3 Vane Condition

The vanes were in excellent condition with no burning noted on any of them. One vane, immediately behind Stage 1 Shroud No. 17, did show some type of impact damage in the center third of the leading edge (see photograph, Figure 8-15 in Appendix B). None of the cooling holes showed indications of being clogged.

#### 6.2.2.4 Spoolie Spring Washers

During the course of the visual inspections, eight spoolie spring washers were observed to be missing. Seven others were so badly worn that they were easily removed with no effort. All others were also worn to the extent that no more spring was left in them; however, they were still secured in place.

# 6.2.2.5 Nozzle Support

Inspection of the forward face of the support that mates to the Stage 1 nozzle vanes, showed good contact the full 360° circumference. All of the support cooling holes appeared to be open.

Drop checks from the forward face of the aft flange to the forward face of the lugs that support the Stage 1 vane outer hook (Dim. "K" in S/M) were taken at 16 equally spaced locations, starting at 12 o'clock, CW. The dimensions were as follows:

1	4.858"	9	4.863"
2	.4.860"	10	4.861"
3	4.861"	11	4.856"
4	4.867"	12	4.858"
5	4.864"	13	4.857"
6	4.865"	14	4.857"
7	4.869"	1.5	4,858"
8	4.857"	16	4.862**
	Average	4.86	7 **
	•		
	Shop Dim.	4.85	7"/4.861"
	Service Limits	4.85	3"/4.865"

Corresponding dimensions from the CRF aft flange to the aft face of the Stage 1 vane outer hook also average 4.861" (Reference 6.2.1.2, "Drop Dimension - CRF to Stage 1 Vanes"). However, this engine incorporated the 0.020" shim between the Stage 2 support mounting flange and the CRF aft flange. Therefore, the actual gap averaged 0.020".

## 6.2.2.6 Interstage Seal Grooves

The grooves in each of the interstage seal lands were measured at 12; 3, 6, and 9 o'clock positions. Measurements were obtained by rubbing a piece of chalk across the groove, and measuring the resultant protrusion. Following are the results:

	I	L	2	2		3	4	
Location	Width	Depth	Width	Depth	Width	Depth	Width	Depth
12 o'clock	0.126	0.070	0.120	0.062	0.110	0.055	0.100	0.055
3 o'clock	0.135	0.090	0.135	0.075	0.110	0.080	0.105	0.060
6 o'clock	0.135	0.040	0.155	0.055	0.150	0.070	0.136	0.042
9 o'clock	0.125	0.055	0.130	0.055	0.130	0.052	0.125	0.052
Average	0.130	0.064	0.135	0.062	0.125	0.064	0.119	0.052

Note: All readings are in inches.

## 6.2.2.7 Stage 1 and Stage 2 Shroud Radii

The Stage 2 high pressure turbine nozzle assembly was restrained on its aft flange on the fixture normally used for shroud grind, and the entire assembly centered on the inspection table. Stage 1 and 2 shrouds were measured at axial locations approximately 1/2" from LE and 1/4" from TE at each end and at the center of each shroud. Measurements at each of these locations consisted of a diameter check at the 12 o'clock position and runouts relative to this point at each of the other positions.

Stage 1 Shroud Runout Data

	1/	2" from	ĽE	1/4	4" from	TE
<b>01</b> 1	· ·					
Shroud	-			_	_	_
No.	1	2	3	1	2	3
.1 .	0.	4	-1	0	5	5
2	2	3	2	10	9	3
2 3	8	. 10	10	4	14	13
4	11	25	15	18	25	17
4 5	15	12	3	18	25	17
6	18	18	14	16	27	24
7	16	18	14	16	21	20
8 -	19	23	20	24	25	24
9	23	23	15	25	28	22
10	17	13	8	24	25	16
11	8	8	6	16	15	12
12	5	6	4	12	15	13
13	5 3 ·	5	2	12	12	10
14	-2	3	3	11	13	10
15	3	9	14	5	15	20
16	13	19	20	20	25	29
17	13	23	13	25	30	17
18	18	17	15	22	27	22
19	15	15	10	23	23	18
20	13	13	10	12	18	17
21 .	11	10	5	14	19	17
22	8	9	5 5	16	16	11
23	6	6	5	14	13	12
24	6	8	0	13	13	6

All readings are in mils and are positive, unless otherwise indicated.

	Leading	Trai <b>li</b> ng
Diameter at 12 o'clock	33.288	<b>33.</b> 275
Radius at 12 o'clock	16.643	16.632
Minimum Radius	16.641	16.632
Maximum Radius	16.668	16.662
Average Radius	16.653	16.649

Stage 2 Shroud Runout Data

	1/	2" from	LE	1/4	" from	TE
Shroud No.	1	2	3	1	2	3
1	0	0	0	0	1	-3
2	-1	-2	3	1	3	0
3	3	7	8	<b>-</b> 1	7	4
4	6	10	8	4	7	1
5	8	9	7	4	5	1
6	5	9	4	1	6	-7
7	2	7	4	<del>-</del> 7	5	2
8	4	12	10	2	8	6
9	9	15	10	3	8	5
10	7	10	5	4	0	-6
11	1	6	6	-6	-4	-3

All readings are in mils and are positive, unless otherwise indicated.

	Leading	Trailing
Diameter at 12 o'clock	34.595"	34.605"
Radius at 12 o'clock	17.293"	17.300"
Minimum Radius	17.291"	17.293"
Maximum Radius	17.308"	17.308"
Average Radius	17.299"	17.301"

#### 6.2.3 High Pressure Turbine Rotor Assembly

#### 6.2.3.1 General Inspection

An overall visual inspection of the HTPR assembly showed it to be in good condition. No discrepancies were noted in any of the spool parts (disks, shafts, seals, etc.). There appeared to be a heavy Stage 2 blade-to-shroud rub, as evidenced by the discoloration at the tips of all blades, accompanied by tip burrs the full blade width. Except for the slight deposit buildup at the leading edge, convex side, the Stage 2 airfoils were smooth.

Stage 1 blades also exhibited some rubbing, but not as much as Stage 2; no burrs were noted on the blade tips. The typical heavy deposits and roughness were seen on the concave surface of the airfoil, with the convex side being smooth.

Photographs of the HPTR can be seen in Appendix B, Figures 8-17 and 8-18.

The Stage 1 and Stage 2 blade retainer wire seals were in many small pieces. These were removed and returned to Evendale. Inspection of the pieces revealed good contact between the seals and the blades.

# 6.2.3.2 HPTR Blade Airfoil Surface Finish

Three blades from each stage were removed to check the surface finish of the airfoils by use of a profilometer. Measurements were taken on each side at 10%, 50%, and 90% of the blade chord. Following are the results of these inspection checks (RMS  $\mu$  1nch):

		<del></del>	—Conv	vex-		Concave →			
Stage	Blade No.	Fwd	Mid	Aft	Avg	Fwd	Mid	Aft	Avg
1	1 2 3	110 130 150	45 50 50	45 30 50	67 70 83.3	75 50 55	105 160 150	195 150 170	125 120 125
Æ	lverage				73.5				123
2	1 2 3	80 120 130	40 - 40 32	40 31 41	53.3 63.7 67.7	35 40 80	40 31 37	70 60 70	48.3 43.7 62.3
A	\verage				61.5				51.5

# 6.2.3.3 Rotor Blade Tip Measurements

The HPT rotor was set up in a lathe bed and the blades shimmed per the shop manual. Runouts at two axial locations (0.100" from both the leading edge and the trailing edge) of each blade, together with the maximum blade radius of each stage, were taken and recorded. Following are the detailed data.

Stage 1 HPTR Blade Runout Data

No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft
1	11	6	28	8	3	55	10	10	82	12	1 9
2	9	2	29	8	1	56	11	11	83	13	10
3	10	7	30	8	3	57	10	8	84	10	9
4	7	4	31	10	3	58	11	9	85	13	10
5	8	5	32	9	3	59	13	و.	86	10	9
6	5	4	33	10	2	60	11	9	87	11	8
7	9	6	34	9	4	61	13	13	88	10	9
8	7	4	35	7	1	62	11	8	89	10	8
9	12	7	36	8	5	63	14	10	90	9	9
10	10	4	37	9	1	64	12	12	91	8	6
11	6	1	38	9	3	65	10	9	92	9	7
12	8	2	39	12	4	66	12	10	93	13	8
13	13	5	40	11	4	67	13	10	94	10	8
14	13	0	41	7	3	68	12	12	95	16	10
15	11	6	42	11	6	69	16	12	96	11	9
16	7	3	43	11	5	70	13	16	.97	16	12
17	11	2	44	9	<i>'</i> 6	71	12	9	98	9	7
18	9	0	45	10	4	72	11	8	99	18	11
19	11	3	46	10	12	73	12	10	100	10	5
20	10	0	47	15	12	74	11	10	101	18	10
21	11	3	48	7	9	75	12	8	102	10	4
22	9	2	49	11	9	76	10	9	103	19	14
23	16	11	50	9	9	77	13	10	104	10	7
24	6	4	51	9	8	78	11	8	105	14	12
25	10	2	52	11	11	79	11	8	106	7	7
25	9	3	53	11	7	80	12	8	107	12	9 7
27	9	3	54	11	13	81	12	8	108	9	7

0 = 16.569 in. = Maximum Blade Radius

Other readings are in mils and are negative

```
Fwd. Max. = 16.564 in. Min. = 16.550 in. Avg. = 16.558 in. Aft Max. = 16.569 in. Min. = 16.553 in. Avg. = 16.562 in.
```

Stage 2 HPTR Blade Runout Data

No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft	No.	Fwd	Aft
1	10	3	30	-8	2	59	13	4	88	10	5
2	7	4	31	10	1	60	12	6	89	11	5
3	7	2	32	6	2	61	10	3	90	11	7
4	8	. 7	33	11	2	62	12	8	91	14	3
4 Ś	8	1	34	6	1	63	11	3	92	11	4
6	8	5	35	11	2	64	11	6	93	12	5
7	7	0	36	6	1	65	14	5	94	10	6
8	7	4	37	10	1	66	11	3	95	11	4
9	7	2	38	7	1	67	13	3	96	10	4
10	8	6	39	11	2	68	12	6	97	12	3
11	8	1	40	8	2	69	<b>1</b> 4	6	98	9	4
12	7	3	41	<b>1</b> 2	2	70	11	5	99	12	4
13	8	1	42	7	0	71	14	6	100	9	5
14	7	4	43	11	2	72	11	6	101	12	4
15	7	1	44	9	2	73	13	4	102	9	4
16	6	3	45	12	3	74	11	6	103	14	4
17	8	0	46	-8	3	<b>7</b> 5	14	4	104	11	5
18	6	2	47	9	2	76	14	7	105	14	4
19	7	0	48	9	5	77	14	5	106	10	2
20	7	3	49	9	1	78	11	5	107	12	2
21	9	0	50	9	5	79	14	6	108	9	1
22	8	1	51	12	3	80	10	5	109	14	6
23	7	0	52	8	3	81	14	5	110	8	4
24	6	2	53	11	4	82	10	4	111	13	6
25	9	1	54	9	5	83	14	4	112	8	3
26	7	4	55	12	1	84	11	4	113	10	1
27	10	1	56	11	3	85	16	6	114	7	0
28	6	0	57	1.0	3	86	11	6	115	9	1
29	10	1	58	10	8	87	13	5	116	9	5

0 = 17.223 in. = Maximum Blade Radius

Other readings are in mils and are negative.

Fwd. Max. = 17.217 in. Min. = 17.207 in. Avg. = 17.213 in. Aft Max. = 17.223 in. Min. = 17.215 in. Avg. = 17.220 in.

## 6.2.3.4 HPT Blade Clearances

Calculated clearances, as derived from the blade tip measurements and the shroud dimensions (Section 6.2.2.7) were as follows:

Stage No.	B/P in.	Min. in.	Max. in.	Avg. in.	ΔB/P in.
1 (LE)	0.072	0.077	0.118	0.095	+0.023
1 (TE)	0.072	0.063	0.109	0.087	+0.015
1 (Avg)	0.072	0.070	0.114	0.091	+0.019
2 (LE)	0.075	0.074	0.101	0.086	+0.011
2 (TE)	0.075	0.070	0.093	0.081	+0.006
3 (Avg)	0.075	0.072	0.097	0.084	+0.009

## 6.2.3.5 Thermal Shield Seal Teeth

While in the lathe bed, measurements were taken of the HPTR thermal shield seal teeth, in the same manner as taken on the forward shaft seals.

Runout	V1	V2	V3	٧4
1	0	0	0	0
2	0	0	0	0
3	0	0.5	1.0	0
4	0	0.5	1.5	1.5
5	0	0.5	1.5	1.5
6	1.5	2.0	2.0	1.0
7	0.5	1.0	1.0	2.0
8	0	1.5	2.0	2.0
9	1.0	1.5	3.0	2.5
10	1.5	2.5	3.0	3.5
11	2.0	3.0	3.5	2.0
12	2.0	3.0	3.0	2.0

0 = Max. Rad = 13.316 13.238 13.157 13.029

Resultant diameters are as follows:

Maximum	26.632"	26.475"	26.313"	26.056"
Minimum	26.628"	26.471"	26.309"	26.053"
Average	26.630"	26.473"	26.311"	26,055"

## 6.2.3.6 Forward Shaft Seals Dimensions

While in the lathe bed, the maximum radius of each tooth of each forward shaft seal was recorded, together with the runouts at 12 equally spaced locations. In the following tabulations, 0 = maximum tooth radius; all other readings are in mils, and are negative.

		Forward	Shaft F	orward S	eal		
	Runout	G1 ,	G2	G3	G4	Ģ5	G6
	1	1.0	2.0	0.0	1.0	2.0	0
₹	2	0	3.0	1.0	3.0	1.5	2.0
	3	0	1.0	0	2.5	0.5	2.0
	4	0	1.0	0	0	0.5	1.0
	5	1.0	2.0	0	1.0	2.0	1.0
	6	1.5	1.0	1.5	3.0	1.0	2.0
	7	1.0	1.0	0	1.0	0	1.5
	8	0.5	2.0	0	1.0	0.5	1.0
	9	0	1.5	0	1.0	0	1.0
	10	0.5	1.0	0.5	1.0	0	1.0
	11	1.0	0	0	2.0	0.5	1.0
	12	1.5	1.0	0	1.0	0	0.5
0 = Max.	Rad. =	3.952"	4.042"	4.121"	4.200"	4.283"	4.361'
Min.	Rad. =	3.9505"	4.039"	4.1195"	4.197"	4.281"	4.359'
•		Calc	ılated D	iameters			

Maximum	7.904''	8.082"	8.242"	8.399''	8.565"	8.721"
Minimum	7.901"	8.079"	8.241"	8.396"	8.563"	8.719"
Average	7.902"	8.081"	8.242"	8.397"	8.564"	8.720"

#### Shop Manual Dimensions

Maximum	7.909"	8.087"	8.250"	8.410"	8.570"	8.730"
Minimum	7.899"	8.083"	8.246"	8.406"	8.566"	8.726"
MRL*	7.864"	8.048"	8.211"	8.371"	8.531"	8.691"

<sup>\*</sup>MRL = Maximum Repairable Limits

Using stationary seal data from 6.1.3.4, clearances were determined to be:

## <u>B70</u>

	1	2	3	4	5	6
Maximum	0.022"	0.013"	0.013"	0.015"	0.011"	0.013"
Minimum					0.00811	
Average					0.010"	

Overall Average Clearance = 0.013" vs 0.010" stackup of production hardware.

	F	orward S	haft Aft	Seal		
Runout	HI	H2.	Н3	Н4	Н5	Н6
1	8.0	8.0	7.0	6.0	6.0	3.0
2	6.0	6.5	5.0	5.0	6.0	3.0
3	2.0	1.0	1.0	2.0	2.5	2.0
4	1.0	4.0	3.0	4.5	2.0	2.0
5	6.5	8.0	7.0	9.0	7.0	3.0
6	7.0	7.0	6.0	6.5	7.5	6.0
7	5.0	5.0	4.0	2.5	2.5	2.0
8	1.5	3.5	2.5	3.0	2.0	1.0
9	5.5	5.5	4.5	4.5	4.5	2.0
10	2.0	2.5	1.0	2.0	2.5	1.0
11	0	0	0	0	0	0
<u>1</u> 2	4.5	4.5	3.0	4.0	3.0	0.5
Rad. =	5.214"	5.298"	5.378"	5.457"	5.536"	5.614"
	5.206"	5.290"	5.371"	5.448"		

## Calculated Diameters

Maximum	10.425"	10.589"	10.752"	10.907"	11.067"	11.225"
Minimum	10.415"	10.583"	10.745"	10.904"	11.062"	11.222"
Average	10.420"	10.587"	10.749"	10.906"	11.064"	11.224"

## Shop Manual Dimensions

	Maximum	10.417"	10.587"	10.747"	10.907"	11.067"	11.227"
	Minimum	10.413"	10.583"	10.743"	10.903"	11.063"	11.223"
•	MRL*	10.378"	10.548"	10.708"	10.868"	11.028"	11.188"

<sup>\*</sup>MRL = Maximum Repairable Limits

Using stationary seal data from 6.1.3.4, clearances were determined to be:

0 = Max. Min.

	$\underline{\mathtt{B71}}$							
	1	2	3	4	5	6		
Maximum Minimum Average	0.007"	0.010"	0.007"	0.013"	0.024" 0.013" 0.019"	0.011"		

Overall Average Clearance = 0.015" vs 0.010" stackup of production hardware.

#### 6.3 LOW PRESSURE TURBINE SECTION

One of the most important objectives of the NASA-Lewis Diagnostics Program is to evaluate low pressure turbine (LPT) performance restoration. In accordance with this, following the inbound performance test, the LPT module was removed from the engine and disassembled into its major components for various inspection checks, and for cleaning of the blades and vanes.

Upon completion of this activity, the module was rebuilt and reinstalled on the engine. This was followed by a test cell run to measure the change in engine and component performance. While the work was being performed on the LPT section, the rest of the engine was not disturbed. This was done so that any performance changes could be attributed only to the refurbishment of the LPT blade and vane surface finish, and not to some unrelated activity.

During the core analytical teardown, the LPT module was removed from the engine and set aside with no further disassembly. In time it was returned to United Airlines "as is" - together with the engine modules and other hardware.

Following are the results of the LPT module analytical teardown and refurbishment.

## 6.3.1 <u>Turbine Midframes</u>

## 6.3.1.1 General

Visual inspection of the turbine midframe revealed two holes (3/4" dia. and approximately  $1" \times 2-1/2"$ ) in the inner liner aft of Strut No. 2; one hole on each side of the strut (see photograph, Figure 8-18 in Appendix B). Aft of Strut No. 2, there was a 4" circumferential crack in the liner, and another circumferential crack approximately 9" long was observed in the liner

behind Strut No. 3.

The aft outer seal had numerous cracks and several missing pieces. A number of cracks were also noted on the Stage 1 nozzle outer support ring.

No attempt was made to repair or to replace any of the discrepant parts. Any change of hardware at this time could have clouded any subsequent data taken to analyze the LP system per the test plan.

It was the consensus of opinion that none of the noted faults were serious enough to imperil the engine during the remaining planned tests. Therefore, after all the inspection checks were complete, the TMF was reassembled to the package together with all the original hardware "as is." The test program was completed without incident.

After removal of the LPT module during the core analytical teardown, the TMF liner was visually inspected, as viewed from the forward end, and no further crack progression was noted. It was noted during this inspection that the buckling of the liner had caused one EGT T/C probe aspirator hole to be immersed in the cooler TMF liner cavity. Another was partially immersed. Possibly this could have been the reason for the inconsistent EGT indicated readings, recorded during the several tests conducted.

## 6.3.1.2 LPT Pressure Balancing Seal

An eight-point diameter measurement of the LPT pressure balance seal produced the following results:

- 1. 19.054" 5. 19.055"
- 2. 19.049" 6. 19.052"
- 3. 19.053" 7. 19.049"
- 4. 19.049" 8. 19.051"

Average = 19.052"; S/M = 19.050"/19.054"

Average clearance (C27) to the rotating seal (see 6.3.2.2) was calculated to be 0.032" vs 0.031" stackup of new production hardware.

## 6.3.1.3 Stage 1 LPTN Vane Airfoil Surface Finish

The airfoils of the end vanes of two Stage 1 LPT nozzle vane segments

were inspected after removal from the TMF. The profilometer and associated hardware used for these surface-finish checks were supplied by Airline Support Engineering (ASE), Evendale. A typical setup of this equipment can be seen in Figures 8-20 and 8-21 in Appendix B.

Following the inspection, these vanes together with all the other Stage 1 vanes were SWECO-cleaned for two hours. Vane segment S/N B0631, one of the two that had been inspected, was subjected to six more hours of cleaning to determine if a longer cleaning cycle would further improve the surface finish. A recheck of the previously inspected airfoils was then made to ascertain the net improvement. The vanes in S/N B0631 did appear to clean up more than the other; but it is felt that a more detailed test is required to establish the optimum amount of the time required for the cleaning cycle.

For comparison purposes, the results of the Stage 1 vane airfoil surface-finish checks have been grouped with similar data acquired on the vane airfoils in the other stages of the LPT section (see 6.3.3.2, "Airfoil Surface Finish Checks"). The measurements were taken 0.45"/0.50" from the leading edge (LE) and the trailing edge (TE) on each side; tip readings were taken 0.50" below the outer platform.

#### 6.3.2 Low Pressure Turbine Rotor

#### 6.3.2.1 General Inspection

A visual inspection of the LPT rotor assembly showed it to be in excellent condition. No discrepancies were observed on any of the spool parts. Blades were rough and dirty, typical of blades which have this amount of running time. A photograph of the rotor assembly can be found in Appendix B, Figure 8-22, showing it mounted in the lathe bed for the inspection checks.

#### 6.3.2.2 Dimensional Inspections

The LPT rotor was set up in a lathe bed on the No. 6 and the No. 7 journals for radial measurements of the blade tip shroud seal serrations, each stage; of the air seal teeth, each stage; and of the pressure balance seal teeth. The results are as follows:

LPTR Blades

	<del></del>	Forward ——	<del></del>	◀	— Aft	<del></del>
Stage	Max. Rad.	Min. Rad.	FIR	Max. Rad.	Min. Rad.	FIR
1 2 3 4	24.133 24.090 24.093 24.118	24.121 24.084 24.082 24.103	0.012 0.006 0.011 0.015	24.129 24.115 24.106 24.119	24.118 24.109 24.098 24.106	0.011 0.006 0.008 0.013
5	24.120	24.101	0.019	24.113	24.100	0.013

#### Interstage Seals

	<del></del>	–Forward <del>–</del>	<b>→</b>	◄	Aft	<b></b>
1	18.191	18.188	0.003	N/A		
2	18.005	18.001	0.004	18.004	18.002	0.002
3	16.851	16.846	0.005	16.850	16.836	0.014
4	15.579 '	15.575	0.004	15.583	15.580	0.003
5	14.216	14.213	0.003	14.223	14.215	0.008

## Pressure Balance Seal

Tooth			
`		·	
$\mathbf{F} - 1$	9.495	9.491	0.004
-2	9.497	9.493	0.004
-3	9.497	9,493	0.004
-4	. 9.497	9.493	0.004
<del>~</del> 5	9,497	9.493	0.004

## 6.3.2.3 Airfoil Surface Finish Checks (RMS)

After the lathe bed inspections, two blades per stage were removed from the rotor for airfoil surface finish checks. Following these checks, all blades were removed from the rotor and SWECO-cleaned for two hours. The surface finish was then rechecked on the same blades as previously checked.

The following is a tabulation of these surface finish inspections: dirty (D), clean (C), and the differences ( $\Delta$ ). All readings were taken 0.10"/0.15" from LE and TE, each side. Tip readings were taken 0.50" below the blade's outer platform.

-			<del></del>	<del></del> (	Conve	x		<b>←</b>	-Conca	ve
		Condi-	T:	Ĺp	Pi	tch			Pitch	Į.
Stage	S/N	tion	LE	TE	LE	TE	Avg	LE	TE	Avg
1	D1249	D .	140	70	80	75	91	70	100	85
		C	100	45	60	<b>7</b> 5	70	60	80	70
	•	· Δ	40	25	20	0	21	10	20	15
1	B7746	, D	130	65	85	*55	93	55	65	60
		С	85	65	55	*65	68	50	50	50
		Δ	45	0	30	*_	25	5	15	10
2	A1774	D	130	80	65	55	83	55	65	60
		C	100	45	45	35	56	45	55	50
		Δ	30	35	20	20	26	10	10	10
2	A3139	D	140	60	60 .	65	81	60	60	60
		С	75	40	50	50	54	55	60	57
		Δ	65	20	10	15	27	5	0	3
3	Z0168	D	90	75	70	45	70	65	55	60
		C	80	60	55	30	56	45	50	47
		Δ	10	15	15	15	14	20	5	13
3	Z0710	D	90	85	60	60	74	65	70	67
		C	55	45	40	40	45	50	40	45
		Δ	35	40	20	20	29	15	30	22
4	Y6830	D .	90	80	60	75	·76	70	70	70
		С	70	35	25	45	44	45	55	50
		Δ	20	45	35	30	32	25	15	20
4	Y3666	D	90	70	65	60	71	65	70	68
		C	70	50	35	45	50	55	65	60
		Δ	20	20	30	15	21	10	5	7
5	X4917	D	70	50	40	55	54	70	50	60
		С	45	30	25	30	32	30	25	27
		Δ	25	20	<b>1</b> 5	25	21	40	25	32
5	X2121	D	90	60	55	60	66	65	55	60
		C	80	30	40	35	46	60	40	50
_		Δ	10	30	15	25	20	5	15	10
Avg.	Rotor	D					76			65
		C					52			51
		Δ					24			14

Note: \*Evidently one and/or the other reading is in error; both omitted from the averages.

## 6.3.2.4 Rebuild

Following the cleaning and surface finish checks, the blades were reinstalled in the spool per position marks (as identified during disassembly). The rotor was then reassembled to the package for the next series of tests.

## 6.3.3 Low Pressure Turbine Stator Assembly

#### 6.3.3.1 General Inspection

Visual inspection of the LP turbine stator assembly showed it to be in good condition. Rub patterns on the shrouds and seals were typical of those observed in the past. (See photographs in Appendix B, Figures 8-23, 8-24, and 8-25.) Casteone impressions were made of the maximum depth rub pattern visually observed in each casing half for all shrouds and interstage seals. A sketch of each of these is shown in Figure 6-1.

The impressions are in the files of ASE Engineering, and no further action is planned for them unless some future testing in the program indicates a need for further study.

#### 6.3.3.2 Airfoil Surface-Finish Checks

Two vane segments, each stage, were removed and the surface finish of each of the end vane airfoils, each segment, was inspected. The remaining segments were then removed and all vanes were SWECO-cleaned for two hours. The previously inspected airfoils were rechecked to determine the effect of the cleaning.

The following is a tabulation of these surface inspections: dirty (D), clean (C), and the difference ( $\Delta$ ). All measurements were taken 0.45"/0.50" from the leading edge and from the trailing edge, each side. Tip readings were taken 0.050" below the outer platform.

Note: Stage 1 vane data (though part of the TMF module) are included here for ease of comparison with the vanes in the rest of the low pressure turbine assembly.

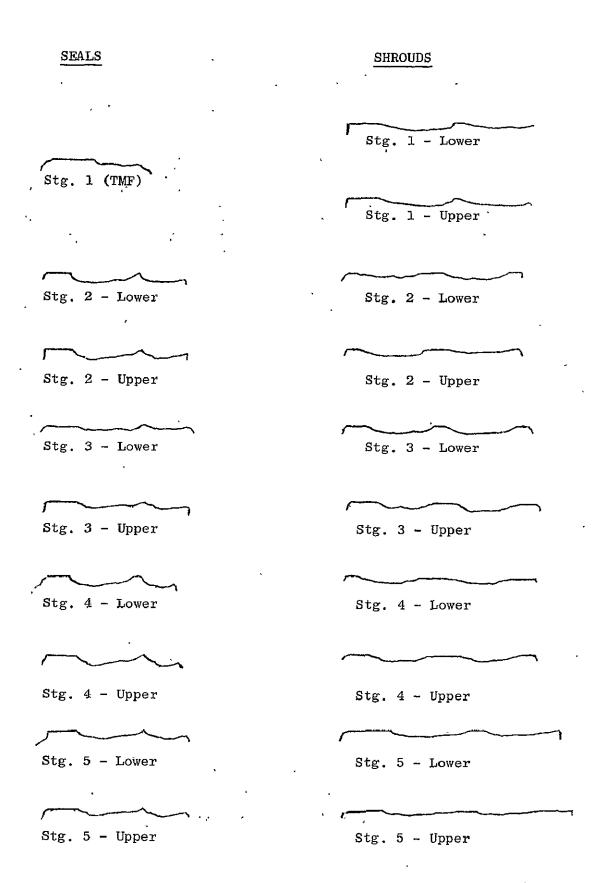


Figure 6-1. LPTS Shroud and Interstage Seal Rub Impressions.

		7	<del></del>		Convex		<del></del>	<b>←</b> C	oncave-	
		Condi-	Ti	.p	]	Pitch		]	Pitch	
Stage	s/n	tion	LE	TE	LE	TE	Avg	LE	TE	Avg
1	в0631	D	120	80	110	90	100	110	110	110
		C	75	35	65	50	56	,60	75	67
		Δ	45	45	45	40	44	50	35	43
1	A9567	D	100	75	95	85	89	90	90	90
		С	70	60	70	45	61	55	60	57
		Δ	30	15	25	40	28	35	30	33
2	B0180	D	150	110	120	90	117	110	105	107
		C	80	50	65	45	60	85	35	60
		Δ	70	60	55	45	57	25	70	47
2	B0164	D	125	120	105	60	102	85	80	82
		С	50	40	50	45	46	66	40	47
		Δ	75	80	55	15	56	30	40	35
3	·T2369	D	90	110	75	65	85	100	75	88
		С	50	55	45	30	45	70	50	60
		Δ	40	55	30	35	40	30	25	28
3	T2157	D	120	100	80	70	93	85	80	82
		С	55	50	40	35	45	65	45	55
		Δ	65	50	40	35	48	20	35	27
4	V1998	D	110	85	70	50	79	65	70	68
		С	60	45	35	20	40	35	35	35
		Δ	50	40	35	30	39	30	35	33
4	U6005	D	130	110	95	65	100	80	100	90
		С	90	70	45	45	62	45	45	45
		Δ	40	40	50	20	38	35	55	45
5	V3698	D	105	95	65	80	86	70	70	70
		С	70	65	35	50	55	45	50	47
		Δ	35	30	30	30	31	25	- 20	23
5	V3647	D	125	85	80	75	91	85	75	80
		C	85	60	50	40	59	50	35	42
		Δ	40	25	30	35	32	35	40	38
Avg.	Stator	D					94			87
		C					53			52
		Δ					41			35

# 6.3.3.3 Rebuild

Upon completion of the cleaning and surface-finish checks, the low pressure turbine stator assembly was reassembled using all the original hardware. The cases were included in the rebuild of the LPT module for resumption of the test plan.

## 6.4 FAN SECTION

Another objective of the NASA-Lewis Diagnostics Program is the evaluation of fan performance restoration with regard to blade leading edge quality and airfoil surface cleanliness. The fan section performance deterioration is believed to be primarily attributed to changes in the fan blade leading edge due to FOD, erosion, etc., and due to buildup of dirt on the airfoil.

To determine the performance effects for the fan section components, the following method was employed: The test cell run after the LPT blade and vane refurbishment served as a baseline for subsequent tests. Following this run, the Stage 1 blades were removed, cleaned, and the leading edges recontoured per Shop Manual. Another test cell run was then conducted. Since no other changes were made to the engine during this time period, the performance improvements achieved were attributed to the fan blade refurbishment. Figure 8-26 in Appendix B compares a recontoured blade with one that had not yet been reworked.

Upon completion of these tests, the core analytical teardown was begun with no further activity on the fan section. Disassembly, as required, was conducted in order to prepare the fan module for shipment back to United Airlines with the other engine modules.

The following are the results of the fan section analytical teardown and refurbishment.

#### 6.4.1 Fan Rotor

When the engine was received from United Airlines, a visual inspection showed 21 Stage 1 fan blades had tiny, insignificant nicks on the leading edges, the majority of which were in the blade outer panel. Otherwise, the blades were in good condition with no large dirt buildups.

Leading edge contour was such that during the test plan, when the blades were reconditioned for test evaluation purposes, only minor rework using Tool 2C7546 was required to bring the contour into limits. See photograph, Figure 8-26, in Appendix B, depicting one blade before and another blade after recontour. The pressure side radius was already in limits, so no hand-blending was necessary on any of the blades.

Visual inspection of the Stage 2 blades, through the Stage 1 fan blades and vanes, showed them to be in excellent condition with no nicks, dents, etc.

#### 6.4.2 Fan Stator

The Stage 1 open faced, aluminum honeycomb shroud exhibited typical ice damage. There were approximately 30 pock marks distributed through the full 360° circumference and located in the path of the blades. The typical size indentation was approximately 3/8" x 1-1/2", with the honeycomb mashed to a depth of 0.050"/0.080". In addition, there were many superficial markings the full width and circumference of the shroud. See photograph, Figure 8-27 in Appendix B, which shows a typical section of the damaged shroud.

The midring shroud (Stage 2), as viewed through the Stage 1 fan blades and vanes, appeared to be in excellent condition. No missing pieces were noted, other than approximately a 1/2" square piece at 1 o'clock. Some light rubs were noted at various locations throughout the circumference, but these are normal. This shroud was still of the abradable material, which has since been replaced by open face, aluminum honeycomb in later production CF6-6 engines and in many updated field engines.

Stage 1 fan blade OGV's also were in excellent condition. There was some slight damage on the leading edge of the aft stator case linings. This too can be attributed to ice damage. A photograph showing the worst observed damage can be found in Appendix B, Figure 8-28.

## 6.4.3 Stage 1 Fan Blade Tip Clearances

## 6.4.3.1 Rotor Runout

The clearance between the shroud and each Stage 1 fan blade was measured at the 6 o'clock position at both the E12 and the E13 locations. The detailed data are as follows:

## Clearances at E12

Blade No.	Clearance	Blade No.	Clearance
1	0.170	20	0.160
2	0.168	21	0.162
3	0.150	22	0.160
4	0.165	23	0.150
5	0.170	24	0.168
6	0.165	25	0.160
7	0.155	26	0.170
8	0.170	27	0.160
9	0.168	28	0.155
1:0	0.165	29	0.170
11	0.160	30 ′	0.163
12	0.160	31	0.152
13	0.170	32	0.163
14	0.165	33	0.160
15	0.170	34	0.167
16	0.155	<b>3</b> 5	0.160
<b>17</b> .	0.165	36	0.145
18	0.165	37	0.180
19	0.165	38 '	0.160
Average Clearance =	0.163"		
Smallest Clearance =	0.145"		
Average E12 Rotor Runout =	0.018" *** 0	.014" Maximum Per B/	<b>σ</b>
Ranout =	0.010 /8 0	OTT MUXIMUM Let. D/	r

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## . Clearances at E13

		•	
Blade No.	Clearance	Blade No.	Clearance
1	0.160	20	0.155
2	0.161	21	0.155
3	0.153	22	0.165
4	0.165	23	0.152
5	0.162	24	0.155
6	0.145	25	0.155
7	0.150	26	0.160
8	0.165	27	0.155
9	0.172	28	0.153
10	0.170	29	0.155
11	0.168	30	0.160
12	0.153	31	0.150
13	0.165	32	0.155
14	0.170	33	0.145
15	0.163	34	0.155
16	0.145	35 ·	0.147
17	0.157	36	0.150
18	0.146	37	0.165
19	0.155	38	0.155
Average Clearance =	. 0.157"		
Smallest Clearance =	0.145"	•	
Average E13 Rotor Runout =	0.012" vs 0	.014" Maximum Per B/	'P

## 6.4.3.2 Shroud Runout

Using the blades with the smallest clearances at E12 (#36) and at E13 (#33) locations, clearances were measured to the shroud at 12 equally spaced locations starting at 12 o'clock and working CW, ALF.

# Clearances at El2 (B/P = 0.145" Min.)

Position No.	Clearance	Position No.		Clearance
12 o'clock	0.150	6 o'clock	٠	0.145
1 -	0.140	7		0.153
2	0.130	8		0.155
3	0.130	9		0.157
4	0.139	10		0.150
5	0.142	11		0.145

Average = 0.145"

# Clearances at El3 (B/P = 0.145" Min.)

Position No.	Clearance	Position No.	Clearance
12 o'clock	0.145	6 o'clock	0.145
1	0.135	7	0.150
2	0.125	8	0.160
3	0.125	9	0.165
4	0.135	<b>`</b> 10	0.163
5	0.145	11	0.155

Average = 0.146"

#### 6.4.3.3 Blade-to-Shroud Clearances

Using the aforesaid information, the Stage 1 fan blade tip clearances were determined to be as follows:

	. <u>E12</u>	<u>E13</u>	<u>B/P</u>
Minimum	0.130"	0.125"	0.145" min.
Maximum Average	0.192" 0.163"	0.192" <sup>'</sup> 0.158"	- 0.163" max.

#### 7.0 ANALYTICAL ASSESSMENT OF PERFORMANCE LOSSES

The 451-479 detailed analytical teardown inspections and measurements were evaluated resulting in a performance stackup using influence coefficients listed in Tables 7-1 and 7-2. The coefficients are based on current "best estimate" of hardware effects on engine performance and may be updated based on information learned during the NASA-Lewis CF6 Jet Engine Diagnostics Program. The performance stackup (Table 7-3) relative to new engine performance, is based on the analytical teardown inspections summarized in the 451-479 engine report. (See Section 6.0.) The first column (assessment) is based on the analytical measurements and influence coefficients, while the second column (measured) is based on the measured test cell performance deltas between the Evendale production test and the Ontario inbound test. (See Section 5.0.)

Note that the core engine stackup (HPC efficiency, HPT efficiency, and parasitics) is significantly different from the measured component analysis. Much of this discrepancy is due to the problems noted in Section 5.0. A slight error in  $A_4$ ,  $A_3$ ,  $A_3$ , or EPR can alter the component assessment significantly.

The 3.23% SFC assessment compares well with the 4.5% measured deltas, which means approximately 72% of the SFC losses have been accounted for. However, the 54° F EGT assessment (as compared to a 108° F measured delta) indicates a problem in evaluating the EGT loss. As stated earlier, the influence coefficients are "best estimates" which may be modified based on the results of the Diagnostics Program. In addition, the analytical analysis obviously does not address to all the possible loss mechanisms. For instance, no method has yet been devised to completely assess the Stage 1 HPT nozzle assembly. Losses due to vane surface-finish deterioration can and are assessed; but beyond that, no influence coefficients are available to cover other detrimental conditions. Conditions such as ballooning, bowing, or burning of vanes, in addition to the size of the gap/interference fit between the Stage 2 HPT nozzle support forward flange and the Stage 1 vane outer flange, cannot be assessed. An excessive gap would result in cooling air

Table 7-1. CF6-6 Influence Coefficients.

Component	Description	°F EGT	% SFC T/O CR.
HPT	,		
Rotor Blades	Surface finish		
Stage 1	rms µ in. for 0.1% nt Suction 26 µ in.	2	0.08 0.06
Stage 2	Pressure 330 Suction 32	2	0.08 0.06 0.08 0.06
Shrouds	Surface finish Tip clearance for 1% nt		
Stage 1 Stage 2	30 mils 50 mils	21 21	0.85 0.62 0.85 0.62
Interstage Seal	20 mils = 0.15% ηt	3	0.12 0.09
Rotating Stationary			
Stator Vanes	Surface finish rms µ in. for 0.1% nt		
Stage 1	Suction 20 Pressure 140	2 2	0.08 -0.06 0.08 0.06
Stage 2	Suction 28 Pressure 240	2 2	0.08 0.06 0.08 0.06
Both Nozzles	None	-	5,00 0,40
Bal. Piston Seal Rotating Stationary	33 mils = 0.1% WCl6 to HP	18	0.72 0.54
LPT			
Nozzles Stage 1 Stage 2 Stage 3 Stage 4 Stage 5			
Rotor Blades	60 µ in. surface finish		
Stage 1	blades and vanes* = 0.41% η2t	3.0	0.31 0.26
Stage 2 Stage 3	= 0.29% n2t = 0.18% n2t	2.1 1.3	0.22 0.18 0.13 0.11
Stage 4	= 0.10% η2t	0.7	0.07 0.06
Stage 5	= 0.02% n2t 1.00% n2t	$\frac{0.1}{7.2}$	$\begin{array}{ccc} 0.01 & 0.01 \\ 0.74 & 0.62 \end{array}$
Shrouds	40 mils tip seal clear		
Stage 1 Stage 2	= 0.28% η2t = 0.20% η2t	2.0 1.4	0.21 0.18 0.15 0.13
Stage 3	= 0.15% η2t	1.1	0.11 0.09
Stage 4	= 0.11% n2t	0.8	0.08 0.07
Stage 5	= 0,06% η2t 0,80% η2t	0.4 5.7	$\begin{array}{ccc} 0.04 & 0.04 \\ 0.59 & 0.51 \end{array}$
Interstage Seals Rotating	20 mils clear		
Stage 1			
Stage 2	= 0.25% n2t	1.8	0.19 0.16
Stage 3 Stage 4	= 0.14% η2t = 0.15% η2t	1.0	0.10 0.09 0.07 0.06
Stage 4 Stage 5	= 0.13% η2t = 0.05% η2t	0.7 0.4	0.04 0.03
	0.54% n2E	3.9	0.40 0.34

\*Pressure (concave) surface values weighted at 1/4 Suction (convex) surface values weighted at 3/4

Table 7-2. CF6-6 Influence Coefficients.

Component	Description	° F EGT	T/O CR.
Interstage Seals Stationary Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	<b>,</b>		
Bal. Piston Seal Rotating Stationary	51 mils = 0.1% WC16 to LP from HP	2	0.25 0.2
CDP SEALS			
Fwd. Seal Rotating Stationary	19 mils = 1% WC16 to HP	18	0.72 0.54
Aft Seal Rotating	33 mils = 1% WC16 to HP	18	0.72 0.54
Stationary			
COMPRESSOR-ALL PARTS Rotor Blades	Dirt buildup, damage, L/E irregularity Tip Clear avg. 10 mils tighter throughout compressor = 1% nc		
	Breakdown - 10 mils each stage:		
	Blade to case: Stage 1-4 0.05% ης Stage 5-16 0.49% ης	1 9.3	0.04 0.03 0.37 0.25
	Vane to spool: Stage 3-7 0.13% nc Stage 8-15 0.33% nc Total 1.00% nc	2.5 6.3 19.0	0.10 0.07 0.25 0.18 0.75 0.54
	Surface finish: 6 rms µ in. = 0.1% nc	2	0.08 0.03
	33% of blades eroded on each stage, Stage 5 on back = 0.7% nc 50% = 1.0% nc		
Compressor Casings	Leaking variable stator bushings		
Stator Vanes	Surface finish: 10 rms µ in. = 0.1%	2	0.08 0.03
FAN			
Vanes	Surface finish:		, ,
Stage 2 OGV - Inner OGV - Outer	87 rms \( \mu \) in. = 0.1% \( \text{nf} \) 87 rms \( \mu \) in. = 0.1% \( \text{nf} \) 80 rms \( \mu \) in. = 0.1% \( \text{nf} \)	0.6	0.07 0.05 0.07 0.05 0.07 0.05
Fan Rotor Blades	Tin alasm 25		
· Stage 1	Tip clear 35 mils - 0.6% nf Surface finish:		0.42 0.30
Stage 2	27 rms μ in. = 0.1% ηf Tip clear 40 mils = 0.3% ηf		0.07 0.05 0.21 0.15
	22 rms μ in. 0.1% ηf	0.6	0.07 0.05

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Table 7-3. Analytical Assessment of 451-479 Losses.

	Assessment			Measured	
	7	EGT	SFC	EGT	SFC
HP Compressor	0.74%	14° F	0.55%	4° F	0.2%
Blade Surface Finish (18 RMS $\mu$ in.) Vane Surface Finish (34 RMS $\mu$ in.) Rotor Land Coating (10 mils)	0.30 0.34 0.10				
HP Turbine	1.22%	. 26° F	1.04%	99° F	4.2%
Stage 1 Nozzle Surface Finish Stage 1 Blade Surface Finish Stage 2 Blade Surface Finish Stage 1 Blade Tip Clearance (+ 19 mils) Stage 2 Blade Tip Clearance (+ 9 mils) Burned Stage 1 Nozzles (9 segments) Stage 1 Shroud Roughness	0.10 0.06 0 0.63 0.18 0.20				
Parasitics	0.24%	4° F	0.17%	0	0
Aft CDP Seal (+ 3 mils-Rotating) Balance Piston Seal (+ 5 mils-Stationary) Forward CDP Seal (0) LPT Pressure Balance Seal (+ 1 mil)	0.09 0.15 0				
LP System	2.03%	10° F	1.47%	5° F	0.1%
Rotor Clearance	0.10%	0.7° F	0.07%		
Stage 2 (+ 14 mils) Stage 3 (+ 7 mils) Stage 5 (+ 3 mils)	0.07 0.02 0.01				
I/S Seal Clearance	0.10%	0.7° F	0.07%		
Stage 2 (+ 4 mils) Stage 3 (+ 4 mils) Stage 4' (+ 1 mil) Stage 5 (+ 8 mils)	0.05 0.03 0 0.02				
Blade Airfoil Surface Finish	0.57%	4° F	0.42%		
Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.29 0.15 0.07 0.05 0.01				
Vane Airfoil Surface Finish	0.56%	4° F	0.41%		
Stage 1 Stage 2 Stage 3 Stage 4 Stage 5	0.23 0.21 0.07 0.04 0.01				
Stage 1 Fan Blade LE Cleanliness	0.70%	0° F	0.50%		
Total		54° F	3.23%	108°	F 4.5%

See Figure 7-1 for Engine Cross Section.

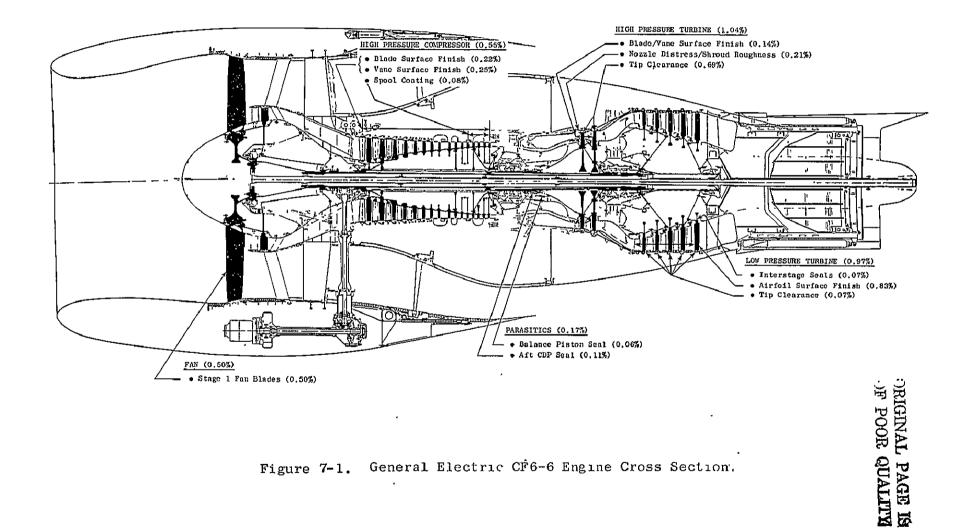


Figure 7-1. General Electric CF6-6 Engine Cross Section.

leakage between the flanges; whereas an extreme interference fit would cause the vanes to be tilted forward, resulting in uneven loading, which also would allow leakage between the flanges. During the Performance Restoration Program for 451-337 in early 1975, back-to-back engine tests were conducted to compare the original hardware versus a new Stage 1 HPT nozzle assembly. A 1.2% improvement in SFC was realized; however, it should be noted that the new Stage 1 HPT nozzle assembly incorporated shims to reduce vane to Stage 2 support interference and the Stage 2 HPTN support flange was reworked to 63 RMS finish. Calculations during the buildup revealed a one mil average interference between the nozzle vane outer flange and the Stage 2 support. Effort is planned as part of this program at a later date to address to this condition.

The Stage 1 fan blades with regard to blade leading edge contour and airfoil surface cleanliness is another example of hardware that cannot be analytically assessed as to performance loss. For this reason, the testing sequence included back-to-back tests comparing performance levels of the blades in the "as-received" condition vs performance levels of the blades after cleaning the airfoils and reworking the leading edges. An 0.5% improvement in SFC was demonstrated and is included in the analytical assessment of losses (Table 7-3).

Other potential areas that do not lend themselves to assessment include the dirt buildup on the HPC airfoils and leakage paths throughout the engine (variable stator bushings, split line flanges, and piping flanges).

The Test Program also included back-to-back tests comparing low pressure turbine performance with blade and vane airfoils in the "as-received" condition versus the same blades and vanes after having been cleaned by the SWECO method. (See Section 6.3.) Airfoil surface finish of two each blades and vanes (each stage) were measured both before and after cleaning, as recorded in 6.3.2.3 and 6.3.3.2.

The analytical assessment of the losses caused by surface-finish deterioration, as compared to original manufacturing requirements, is contained in Table 7-3. Table 7-4 shows the assessment of the performance recovered by

Table 7-4. Analytical Assessment of Refurbished Airfoils.

	<u> </u>	EGT	SFC
Blade Airfoil Surface Finish	0.35%	2.5° F	0.26%
Stage 1	0.14		
Stage 2	0.10		
Stage 3	0.06		
Stage 4	0.04		
Stage 5	0.01		
			a
Vane Airfoil Surface Finish	0.69%	5.0° F	0.51%
Stage 1	0.25		
Stage 2	0.25		
Stage 3	0.12	•	
Stage 4	0.06		
Stage 5	0.01		
Total	1.04%	7.5° F	0.77%

cleaning of the airfoils. As can be seen, 0.83% SFC loss was assessed and 0.77% was regained; or, in other words, 93% of the SFC assessment for airfoil surface finish deterioration was recovered as a result of the refurbishment. These figures tend to substantiate the cleaning method; however, more sampling should be undertaken to further prove the process. The measured improvement due to cleaning the airfoils, however, was negligible as reported in Section 5.0. This tends to indicate that the LPT airfoil surface finish influence coefficients must be reevaluated since an 0.8% improvement in LPT efficiency was expected.

When comparing the analytical assessment to the measured performance deterioration, it must be realized that some of the designated hardware deterioration may have occurred prior to running the Evendale performance acceptance test. The seal break-in run and engine accels may cause some of the seal and blade clearances to open prior to running the performance test. For the purpose of this analysis, however, it is assumed that all the losses occur after the official production acceptance test.

# 8.0 APPENDICES

- 8.1 APPENDIX A FUEL ANALYSIS
- 8.2 APPENDIX B PHOTOGRAPHS
- 8.3 APPENDIX C LOG SHEETS, INBOUND RUN
- 8.4 APPENDIX D LOG SHEETS, TESTS 2 AND 3.

# 8.1 APPENDIX A - FUEL ANALYSIS

# GENERAL 🚳 ELECTRIC

	. MAIL DROPM82_
EVENDALE PLANT CINCINNATI, OHIO 45215  • SUBJECT	DIAL COMM 8 • 332 3743
FUEL ANALYSIS	COPIES:
May 5,1977	•
J. Smith	
H.D. F117	•
All from that 400 field that year way property property are	
Following are the results of fue Lubes Lab	el analysis by the Bearings/Gears & Fuels/
Sample Identification:	1. "Sample #1" (Composite of two) CF6-6D #451-4 2. "Sample #2" (Composite of three) " 3.
Hydrogen:	1. 1395% 2. 1389% 3.
Sulfur:	1. 0/36°/0 2. 0/36°/0 3/
Net Heat by Precision Bomb:	1. 18602 Btu/# 2. 18600 Btu/#
Comments Specific que	isty.
	50mpk #1 , 7885 @ 60°F

Jack F. J. M. Fausz

## 8.2 APPENDIX B - PHOTOGRAPHS

The photographs listed below are included in this report and were selected to support discussions. Other photographs, not included in this report, are available in the CF6 Diagnostics Field Engineering files.

## Figure

- 8-1. High Pressure Compressor Rotor (Cleaned), Missing Blades at Surface Finish Check.
- 8-2. HP Compressor Rotor Stages 13 to 15 Land Coating.
- 8-3. HP Compressor Stator Upper Overall Oil, Dirt, Rubs.
- 8-4. HP Compressor Stator, Upper Stage 7 to OGV Rubs.
- 8-5. HP Compressor Stator, Lower Stage 9 to OGV Rubs.
- 8-6. Combustor, Overall Cracks.
- 8-7. Combustor, Closeup Crack at 7 o'clock.
- 8-8. Compressor Rear Frame, Forward Seals Overall View, Installed.
- 8-9. No. 4B Pressure Balance Seal, Overall View.
- 8-10. Stage 1 HP Turbine Nozzle Assembly, Aft End, Overall View.
- 8-11. Stage 1 HP Turbine Nozzle Assembly, Trailing Edge, Burnt/Distorted Vanes.
- 8-12. Stage 1 HP Turbine Nozzle Assembly Distress, Aft End.
- 8-13. Stage 1 KP Turbine Nozzle, Typical Vane Distress.
- 8-14. HP Turbine Nozzle Assembly, Overall View, Forward End.
- 8-15. Stage 2 HP Turbine Nozzle, Damage Vane/Stage 1 Shroud.
- 8-16. Stage 2 HP Turbine Nozzle Stage 2 Shroud Rub.
- 8-17. HP Turbine Rotor, Overall View of Blades.
- 8-18. HP Turbine Rotor, Blade Tip Rubs.
- 8-19. Turbine Midframe Liner Distress.
- 8-20. Profilometer Setup, LP Turbine Nozzle Segment, Concave Side.
- 8-21. Profilometer Setup, LP Turbine Nozzle Segment, Convex Side.
- 8-22. LP Turbine Rotor, Overall View.
- 8-23. LP Turbine Stator Assembly, Overall View, One Casing.
- 8-24. LP Turbine Stator Assembly, End View of Shroud and Seal Rubs.
- 8-25. LP Turbine Stator Assembly, Stage 1 Shroud Rub Pattern.
- 8-26. Stage 1 Fan Blades Before/After Recontour.
- 8-27. Stage I Fan Shroud Ice Damage.
- 8-28. Fan Stator Aft Liner Ice Damage.

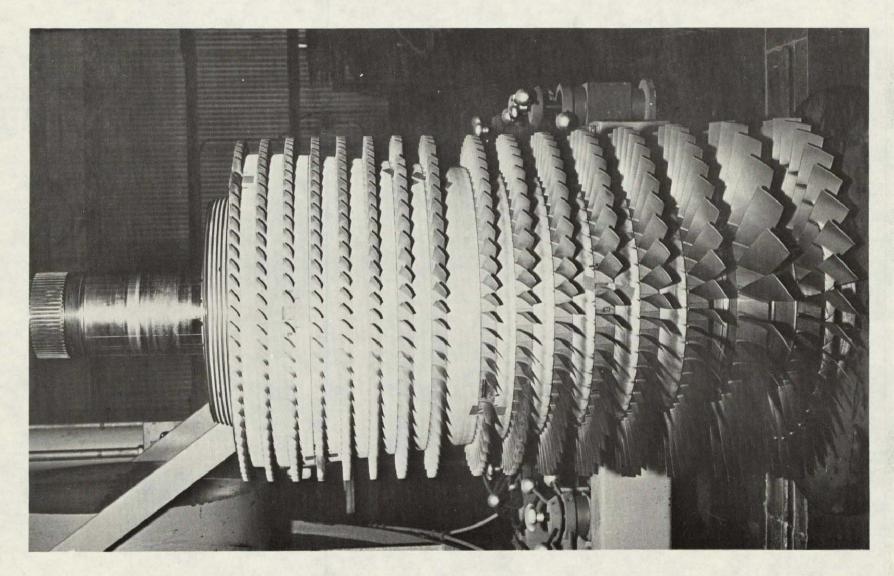


Figure 8-1. High Pressure Compressor Rotor (Cleaned), Missing Blades at Surface Finish Check.

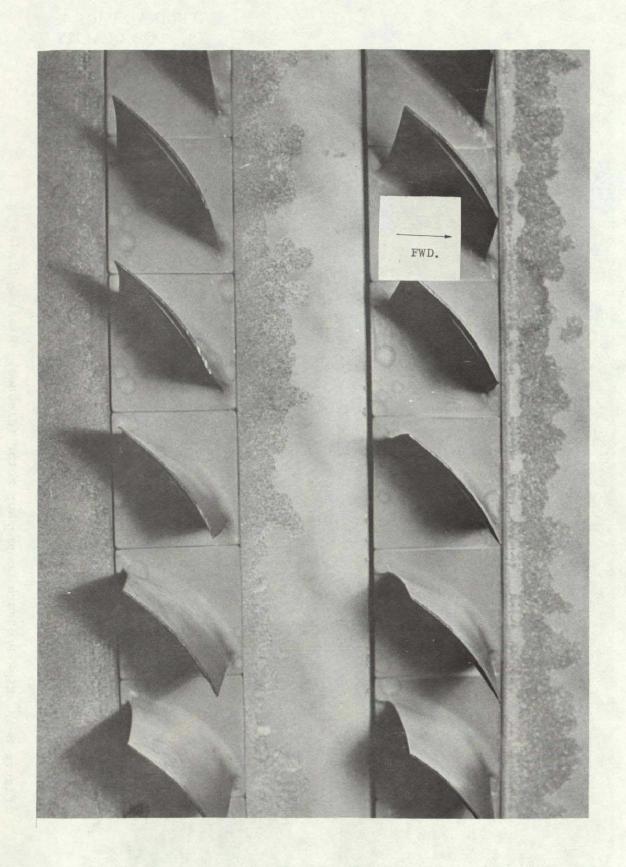


Figure 8-2. HP Compressor Rotor Stages 13 to 15 Land Coating.

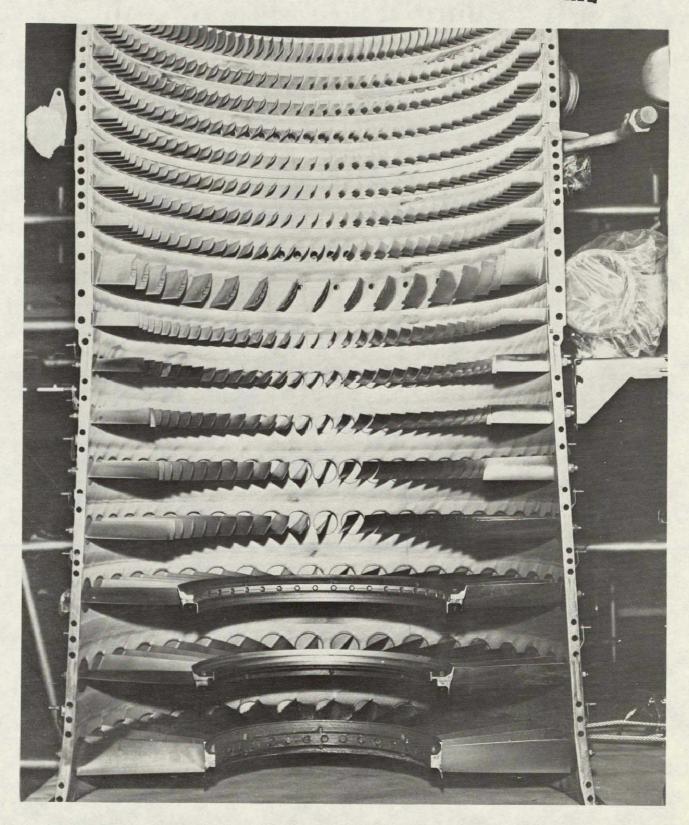


Figure 8-3. HP Compressor Stator Upper Overall Oil, Dirt, Rubs.

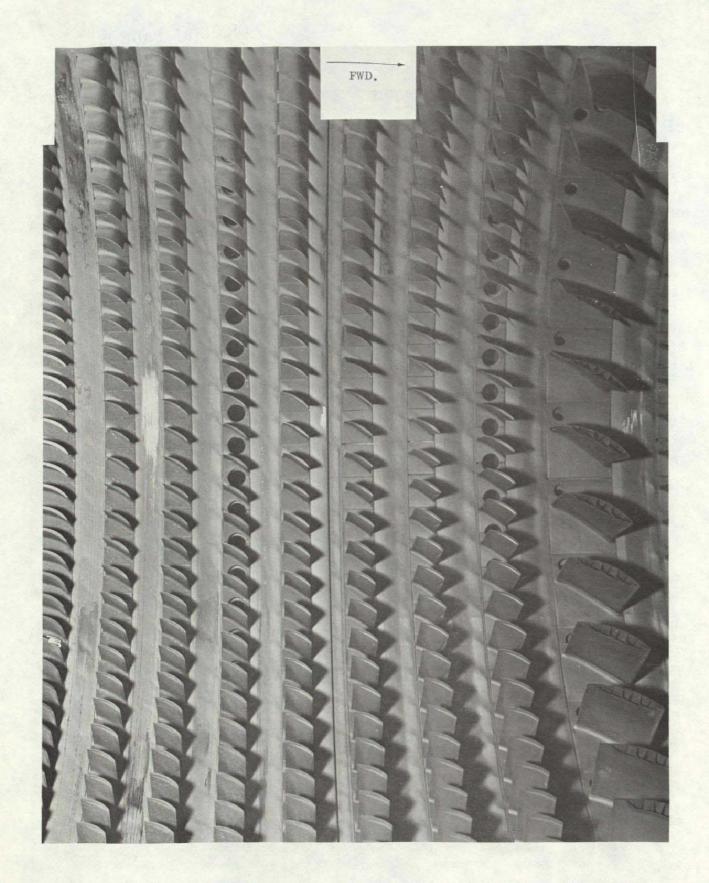


Figure 8-4. HP Compressor Stator, Upper Stage 7 to OGV - Rubs.

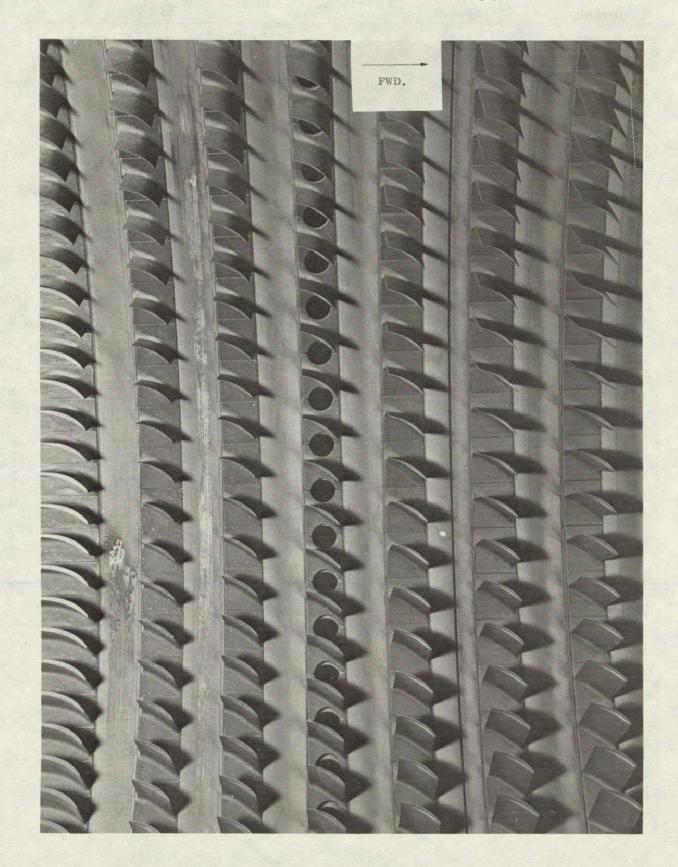


Figure 8-5. HP Compressor Stator, Lower Stage 9 to OGV - Rubs.

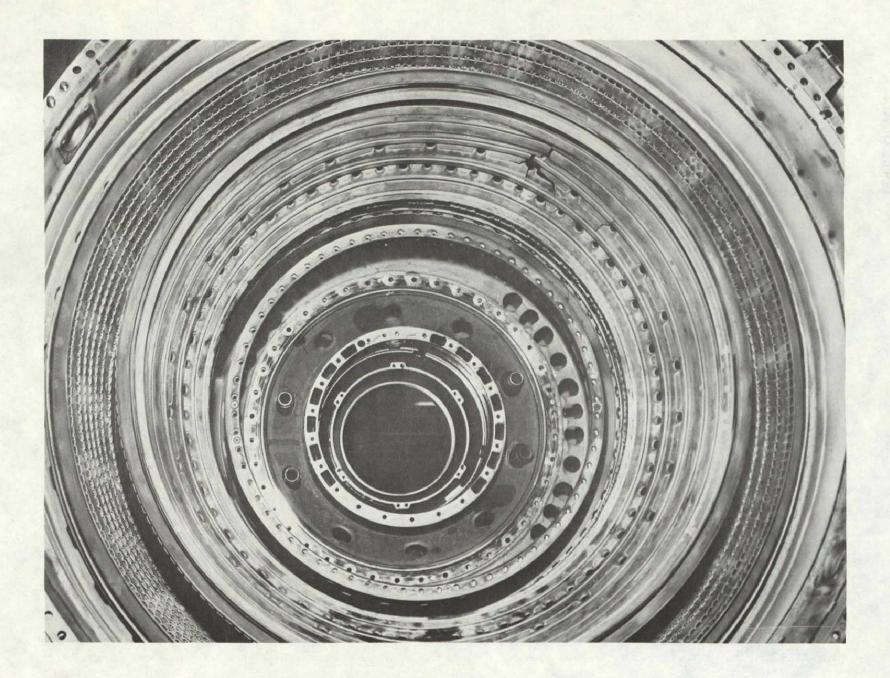


Figure 8-6. Combustor, Overall Cracks.

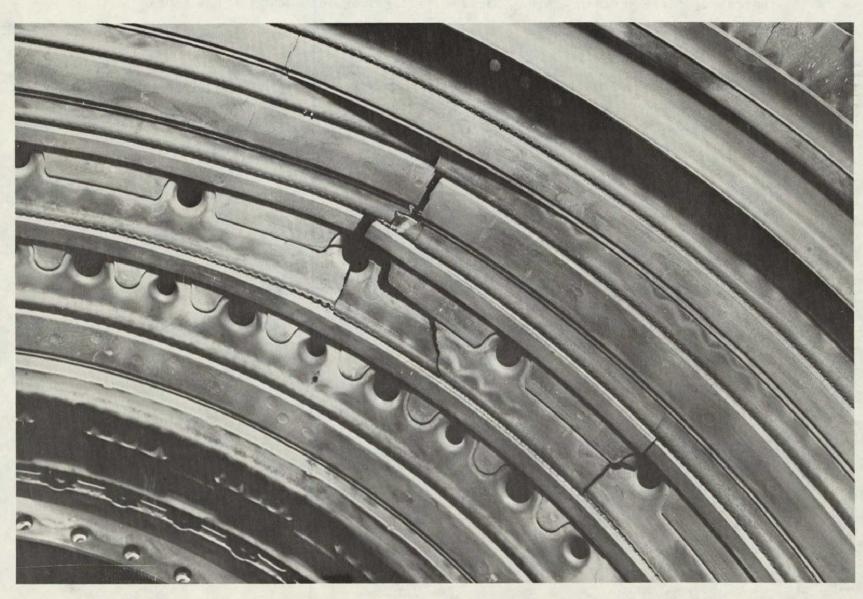


Figure 8-7. Combustor, Closeup Crack at 7 o'clock.

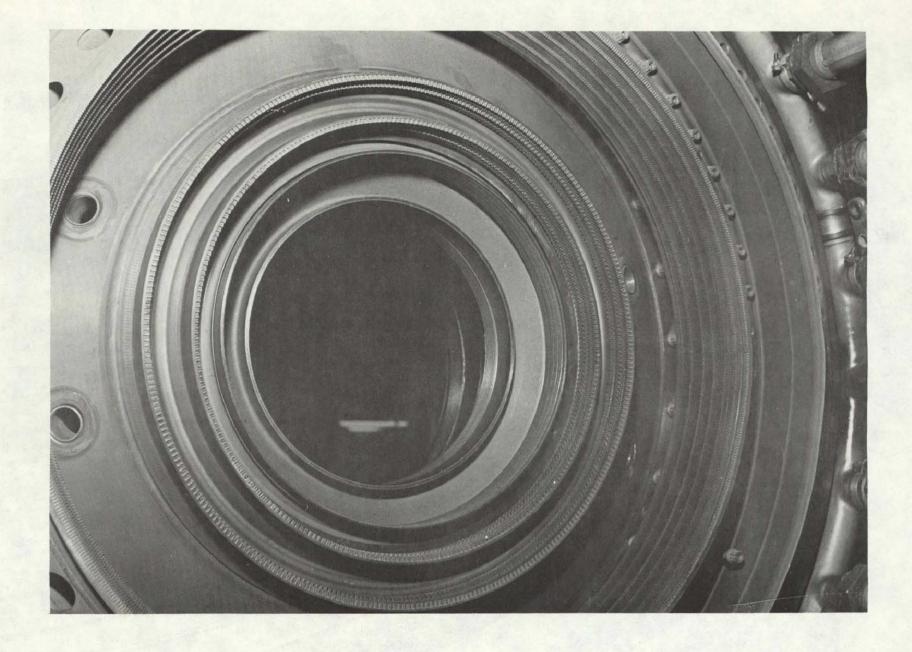


Figure 8-8. Compressor Rear Frame, Forward Seals - Overall View, Installed.

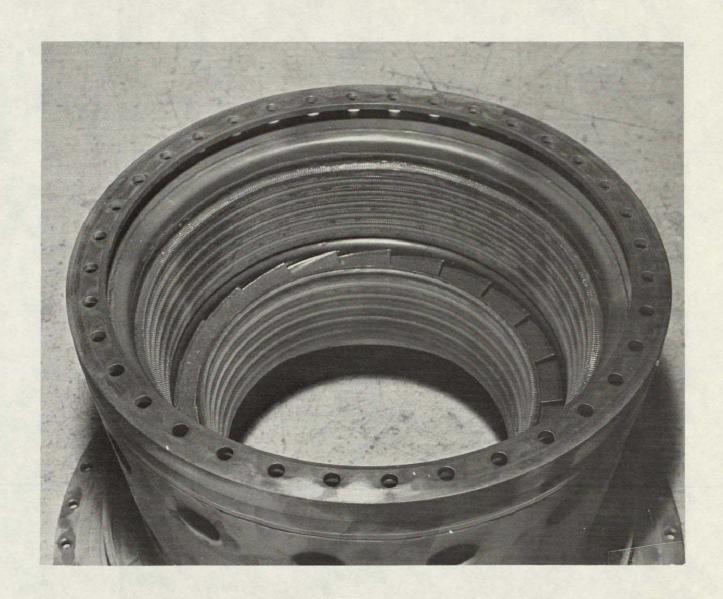


Figure 8-9. No. 4B Pressure Balance Seal, Overall View.

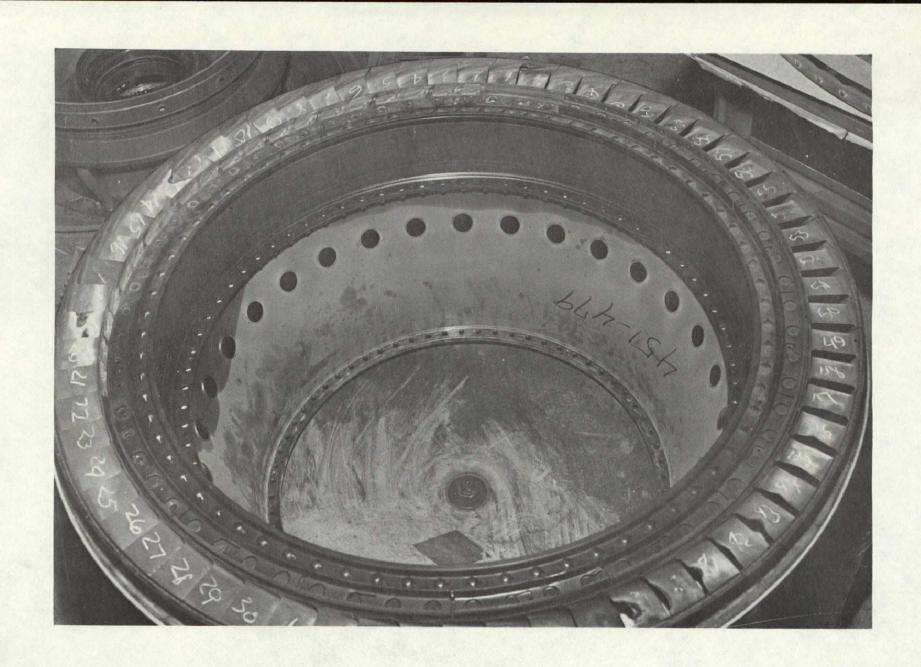


Figure 8-10. Stage 1 HP Turbine Nozzle Assembly, Aft End, Overall View.



Figure 8-11. Stage 1 HP Turbine Nozzle Assembly, Trailing Edge, Burnt/Distorted Vanes.

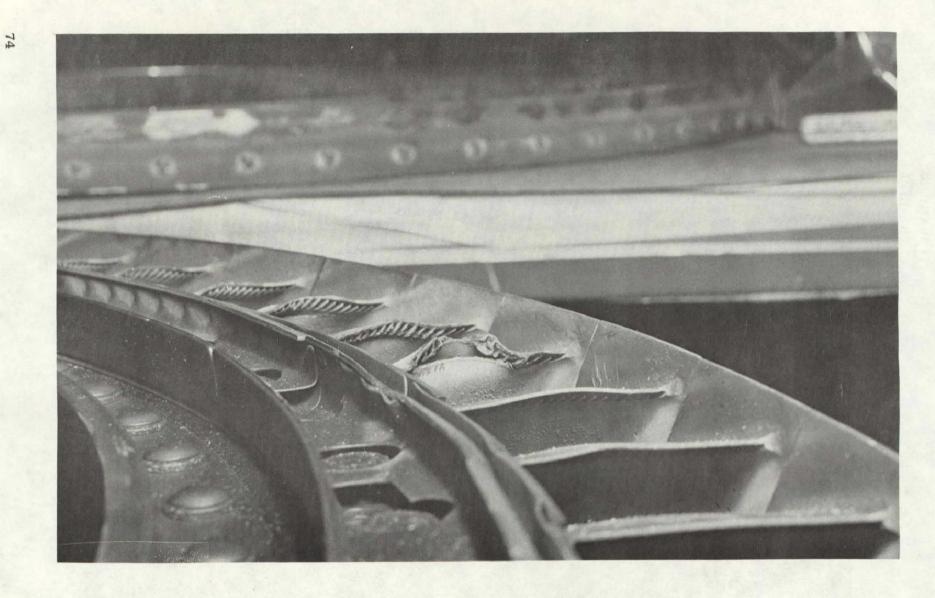
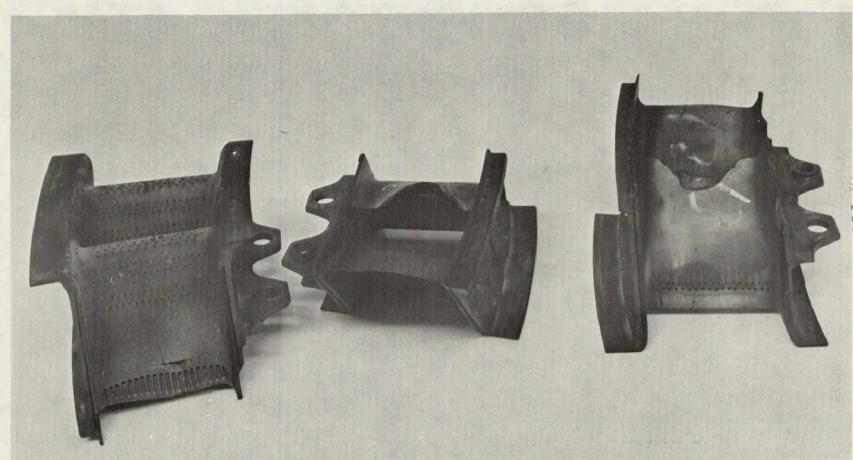


Figure 8-12. Stage 1 HP Turbine Nozzle Assembly Distress, Aft End.



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Figure 8-13. Stage 1 HP Turbine Nozzle, Typical Vane Distress.

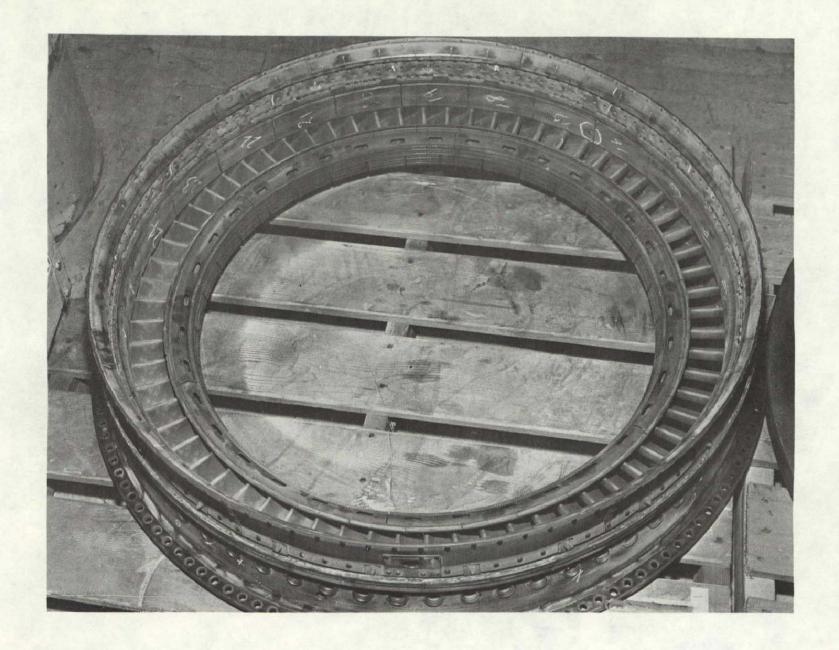


Figure 8-14. HP Turbine Nozzle Assembly, Overall View, Forward End.

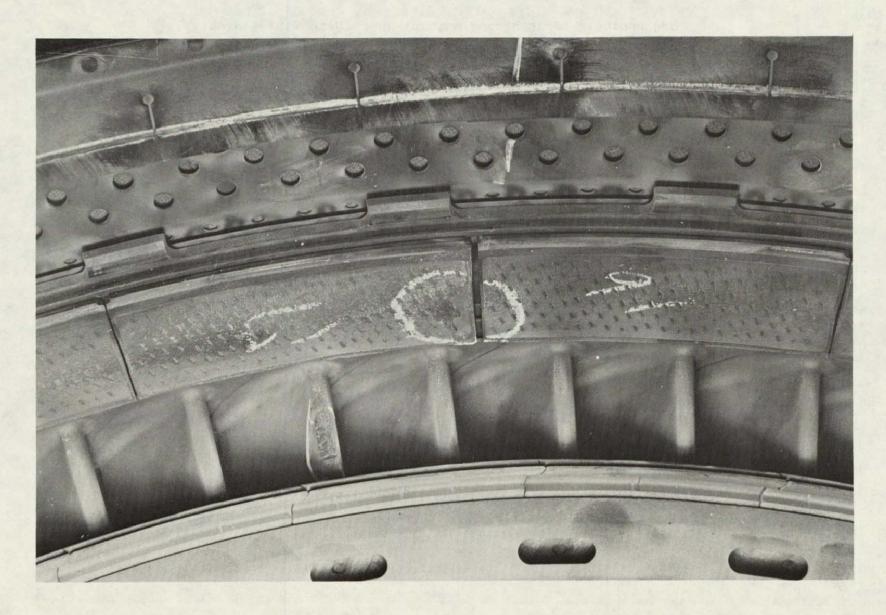


Figure 8-15. Stage 2 HP Turbine Nozzle, Damage Vane/Stage 1 Shroud.

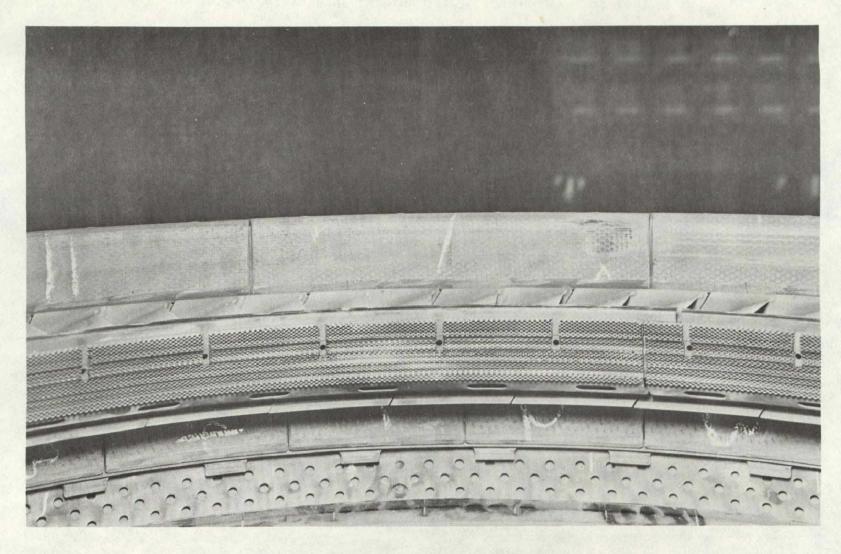


Figure 8-16. Stage 2 HP Turbine Nozzle Stage 2 Shroud Rub.

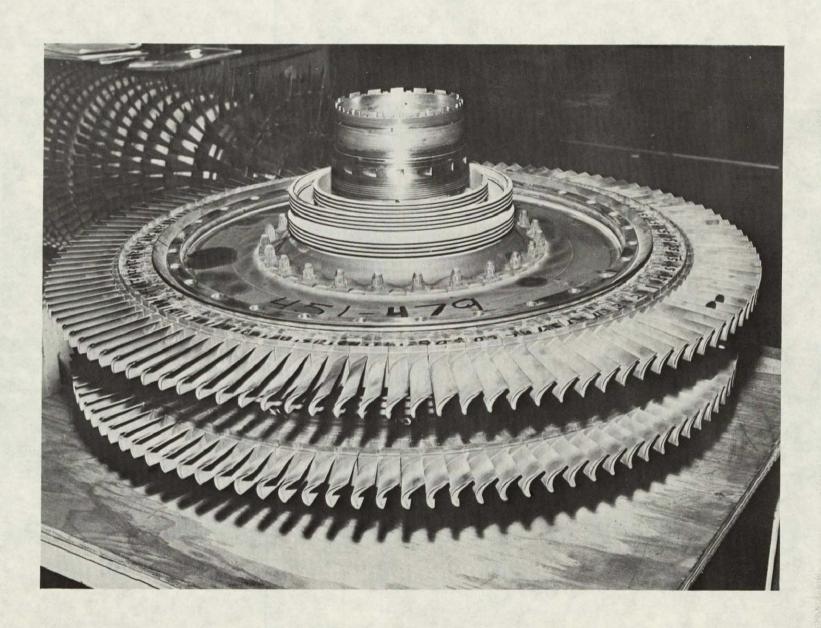


Figure 8-17. HP Turbine Rotor, Overall View of Blades.

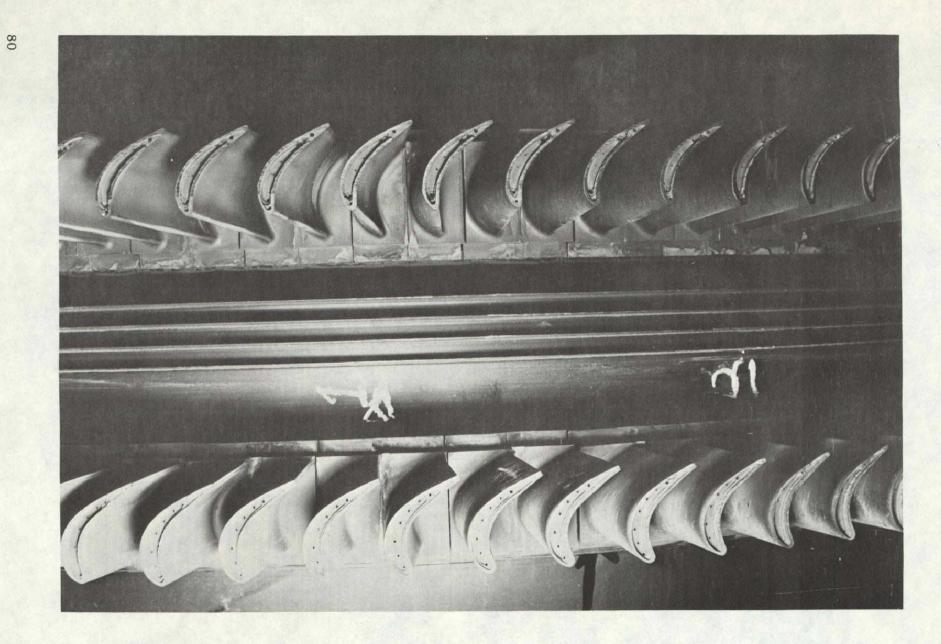


Figure 8-18. HP Turbine Rotor, Blade Tip Rubs.

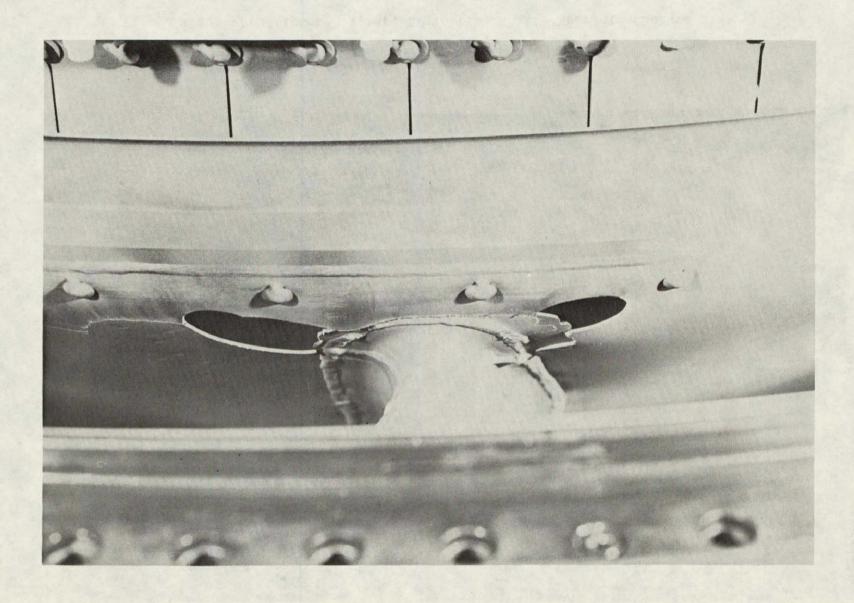


Figure 8-19. Turbine Mid Frame Liner Distress.

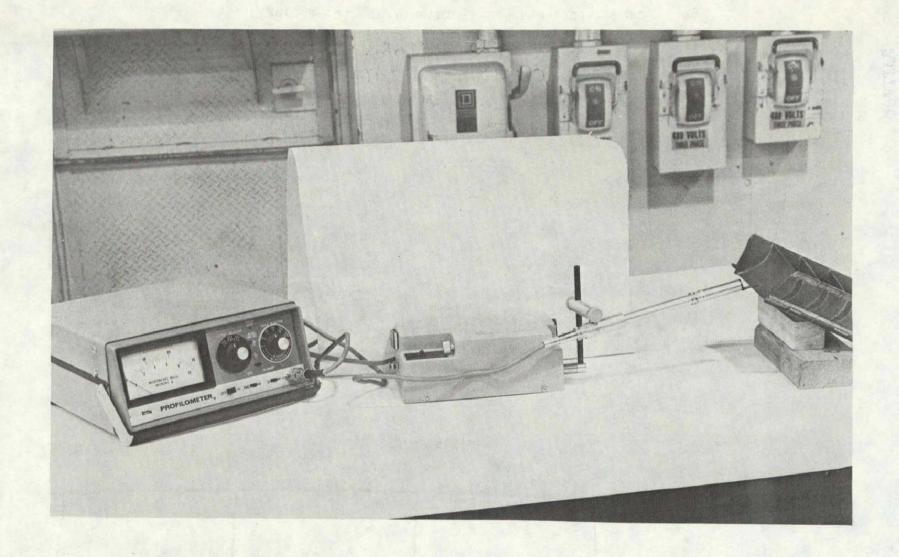


Figure 8-20. Profilometer Setup, LP Turbine Nozzle Segment, Concave Side.

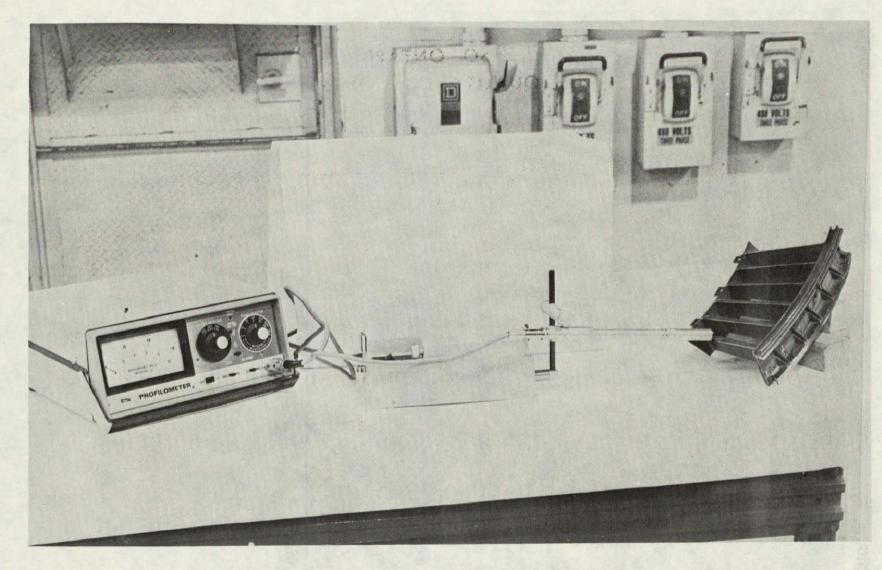


Figure 8-21. Profilometer Setup, LP Turbine Nozzle Segment, Convex Side.

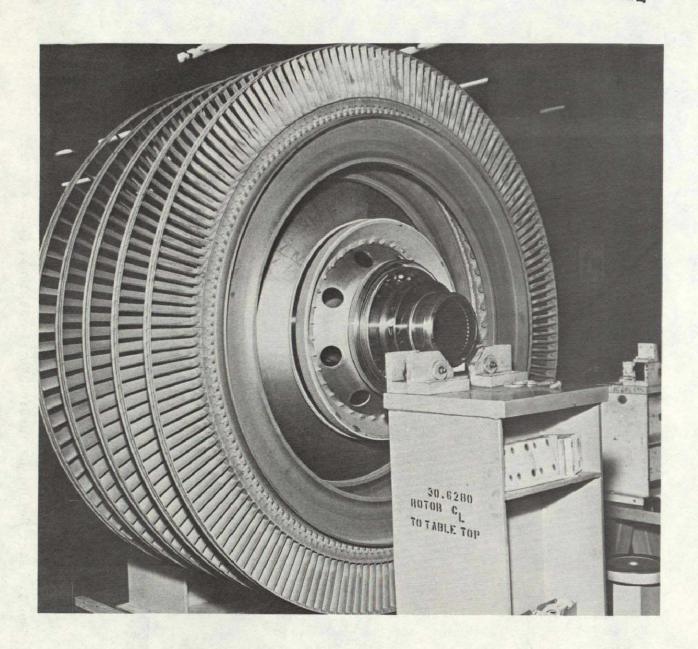


Figure 8-22. LP Turbine Rotor, Overall View.

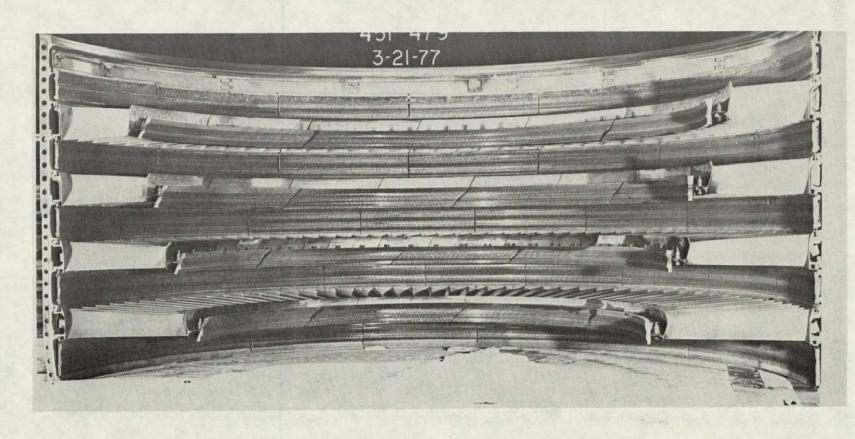


Figure 8-23. LP Turbine Stator Assembly, Overall View, One Casing.

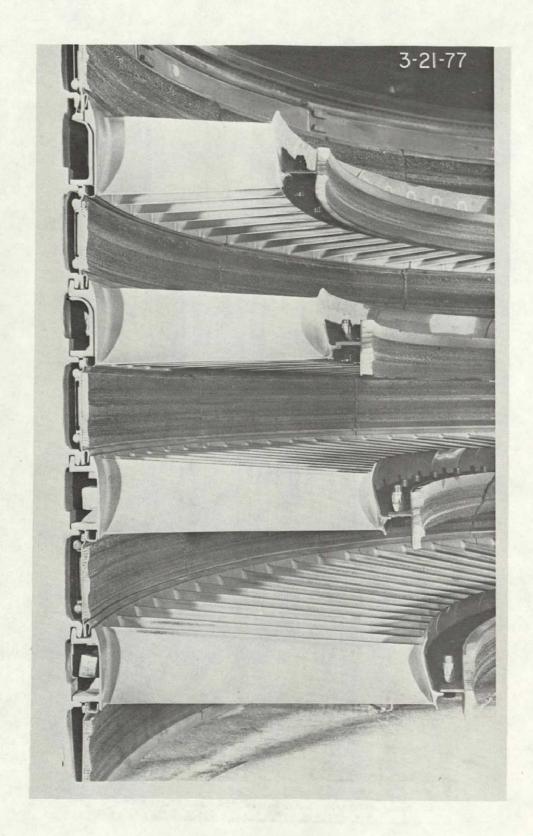


Figure 8-24. LP Turbine Stator Assembly, End View of Shroud and Seal Rubs.

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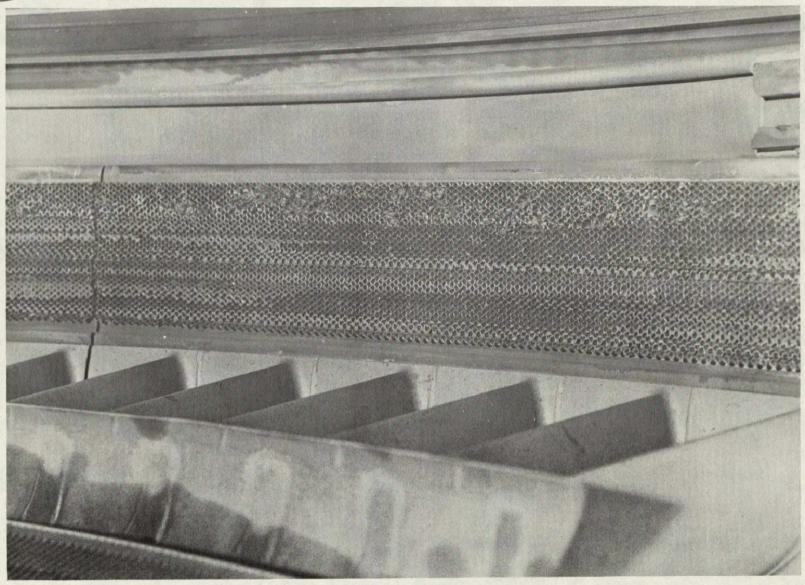


Figure 8-25. LP Turbine Stator Assembly, Stage 1 Shroud Rub Pattern.

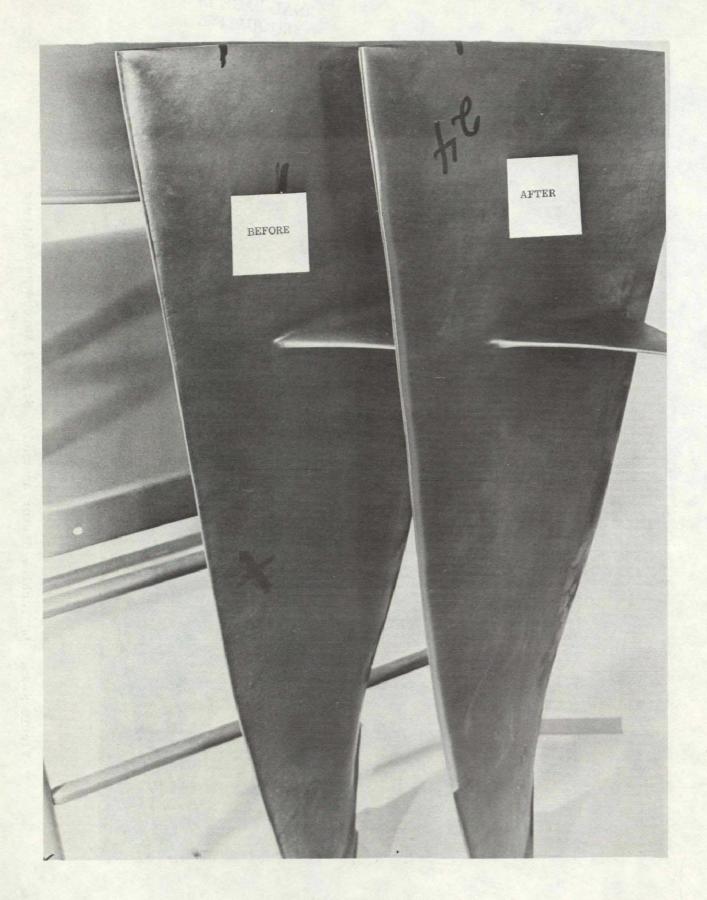


Figure 8-26. Stage 1 Fan Blades Before/After Recontour.

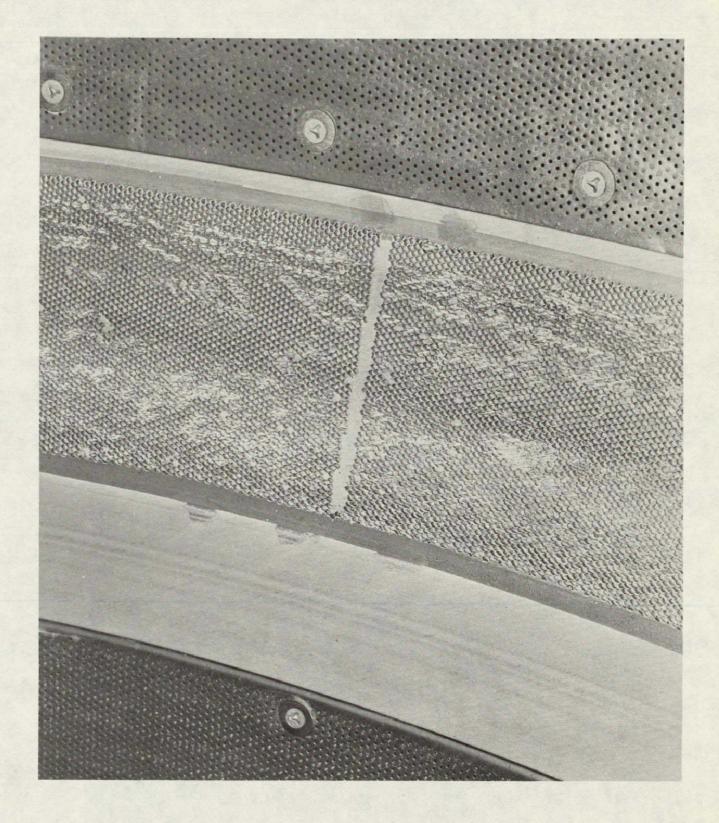


Figure 8-27. Stage 1 Fan Shroud Ice Damage.

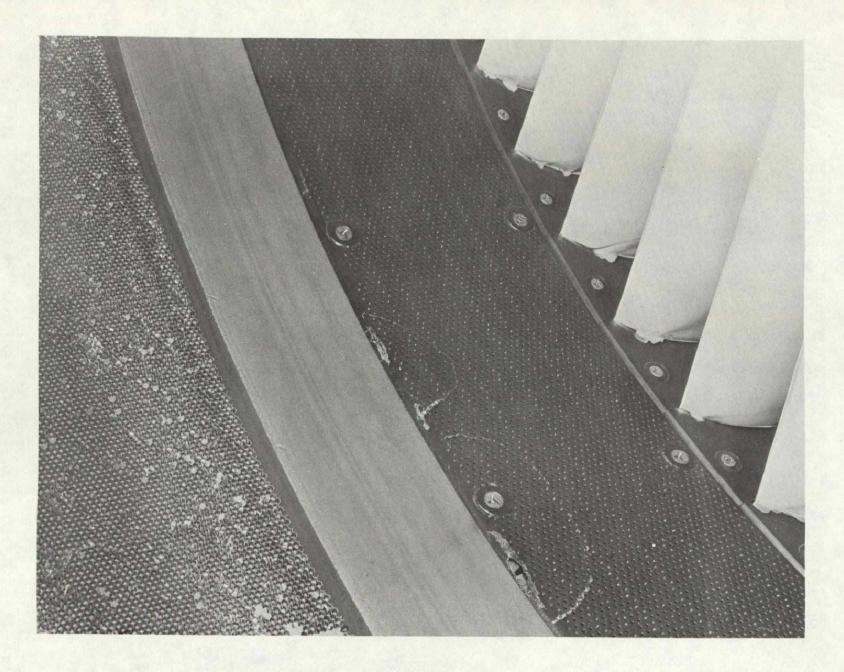


Figure 8-28. Fan Stator Aft Liner Ice Damage.

8.3 APPENDIX C - LOG SHEETS, INBOUND RUN

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## 8.4 APPENDIX D - LOG SHEETS, TESTS 2 AND 3

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