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ANALYSIS AND DESIGN OF A HIGH TIP SPEED, LOW SOURCE NOISE AIRCRAFT FAN INCORPORATING SWEPT LEADING EDGE ROTOR AND STATOR BLADES

Richard E. Hayden Donald B. Bliss Bruce S. Murray K.L. Chandiramani Joseph I. Smullin with Pierre G. Schwaar (AVCO Lycoming Division)

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FOREWORD

The purpose of this report is to describe a recently completed program to design and manufacture an experimental transonic fan model featuring novel methods for noise reduction at the source. The program was conducted between 1974 - 1976 under contract NAS3-18512 issued by NASA Lewis Research Center, with Bolt Beranek and Newman Inc. (BBN) as the prime contractor and AVCO Lycoming as a subcontractor. The contract resulted from a NASA request for proposals (RFP) concerning CTOL aircraft engine fan source reduction concepts. The intent of the RFP was to identify advanced design concepts for reducing both rotor and stator sources which could be implemented with existing aerodynamic and structural design capabilities. The RFP encouraged proposals to reduce noise from high speed single stage fans.

BBN proposed the use of "subsonic leading edge" rotor blades and variably swept stator vanes as the concepts to be investigated. The study and engineering work culminated in the fabrication of a 20-inch diameter fan stage to be tested for acoustic and aerodynamic performance at the Lewis Research Center, National Aeronautics and Space Administration, Cleveland, Ohio.

Bolt Beranek and Newman Inc. (BBN), Cambridge, Massachusetts, served as the prime contractor with overall program responsibility, as well as prime technical responsibility for the fan acoustic design, and other areas. The Lycoming Division of AVCO Corporation, Stratford, Connecticut, was a major subcontractor to BBN, with responsibilities in aerodynamic and mechanical design, and manufacture of the fan hardware. Rotor blades and stator vanes were manufactured under subcontract by New England Aircraft Products, Farmington, Connecticut.

Also included in the program were efforts to develop a 3dimensional compressible flow computer program to analyze the flow through the rotor, especially in the vicinity of the leading edge, and the investigation of the feasibility of using porous trailing edges on the stators to reduce broadband noise. The 3-D flow program was discontinued at the time the rotor design was finalized, and the porous edge concept was not used because of the difficulties perceived in manufacture of small vanes from available porous metal materials.

Numerous individuals at BBN and AVCO made significant contributions to this project. Mr. Richard Hayden served as project manager, and contributed to the acoustic design of the fan as well as other areas. Dr. Donald Bliss served as an associate project manager and had responsibility for the concept of the rotor blade, the rotor acoustic design, and the coordination of the aerodynamic design with AVCO. Mr. Bruce Murray also served as an associate project manager and supervised the mechanical design and manufacturing aspects of the fan. The stator acoustic design was carried out by Dr. K.L. Chandiramani, and Mr. Joseph Smullin. Drs. John McElman and John O'Callahan performed finite element stress analysis of the rotor blades, and Dr. O'Callahan contributed to numerical fluid mechanical analysis of the rotor flow field.

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At AVCO Lycoming, Mr. Pierre Schwaar served as the principal investigator and has primary program responsibility for the fan aerothermodynamical design, and for implementing the subsonic rotor leading edge concept and the acoustic design of the stator blades within operational structural constraints. Mr. Herbert Kaehler led AVCO's work on structural analysis and Mr. John Banks supervised mechanical design and manufacturing activities there.

Mr. James G. Lucas of the NASA Lewis Research Center's V/STOL and Noise Division was the NASA Program Manager, and contributed valuable assistance in the mechanical design and manufacturing areas, and in the integration of the fan into NASA's test facilities.

This report has been designated as Bolt Beranek and Newman (BBN) Report No. 3332.

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SUMMARY

On current generation high bypass ratio turbofan engines, the fan is a predominant noise source which must be controlled to meet future aircraft noise goals. Of the various approaches to turbofan engine noise reduction, the most attractive is reducing the strength of the noise-producing elements at the source, thus avoiding weight and performance penalties associated with various sound suppression approaches.

In modern high bypass ratio turbofans, the fan thrust is achieved in a single fan stage, which usually requires supersonic tip speeds of the fan rotor to produce the necessary pressure rise. In such fans, the predominant sources of noise are shocks radiated from the supersonically-moving rotor blades (called multiple-pure-tone [MPT] noise), and tones radiated from the rotor wake interaction with stator vanes.

In this program, two advanced noise reduction concepts were applied to the design of a 1.6 pressure ratio single stage fan. The goal of the design was to reduce the following acoustic multiple pure tone noise, rotor-wake/stator-blade sources: interaction noise, and noise due to operating the rotor in distorted or turbulent inflow. Unique nonradial blading of the rotor and stator was used to achieve these goals. The rotor blade leading edges were swept so that the normal component of flow to the edge is subsonic at all points along the blade span, thus preventing the occurrence of leading edge shockwaves. The stator vanes were designed to minimize noise generated by rotor wakes incident on the blades by progressivly sweeping the vanes from root to tip in order to produce subsonic trace speeds for the unsteady loads along the span. Special aerodynamic and structural design considerations were required to assure the performance and integrity of this unusual blade and vane design.

The rotor design using a blade concept with shock-free leading edges (except at points of inflection where weak conical shocks occur) is highly flexible in that a large family of blade shapes and leading edge contours may, in general, be used to achieve the noise reduction goal. The swept rotor design is also attractive since it should perform equally well at off-design conditions if it has been designed to perform properly at the highest envisioned rotor speed. The swept edges also are compatible with reducing noise generation due to inflow distortion. In the final design of the particular rotor ultimately constructed, a reversal of the sweep direction was required near mid-span to minimize stress levels in the blade. Once this was done, aeroacoustical-structural design iterations led to a blade with acceptable stress levels and no additional compromise in acoustic performance beyond the expected weak conical shock at the sweep reversal point.

The aerodynamics of subsonic leading edge rotor cascades with supersonic absolute inflow velocities are not well known, and will clearly require further study.

The concept of forcing the trace speeds of moving load distributions on stator vanes to be subsonic was introduced for the first time in this program. The design of a stator which uses this concept requires a controlled rate of axial sweepback (or circumferential skew), the details of which depend heavily on the rotor wake field which varies with distance from the rotor. The selection of a stator vane number for a given rotor design is done with the familiar cutoff condition in mind; however, supersonic rotor tip speeds make it impossible to completely cut off the radiation at the tips of the stator vanes. No serious aerodynamic or structural problems were associated with the swept stator. The stator acoustic design procedure is now well-defined in terms of flow parameters needed as inputs, but the ability to predict the necessary flow parameters of the rotor wake field is presently limited.

SECTION 1

INTRODUCTION

With the advent of high bypass ratio turbofan engines, and the associated decrease in exhaust velocity, the fan stage has become the dominant aircraft engine noise source. Therefore, fan noise reduction is a problem of primary importance in the ongoing effort to evolve quieter aircraft. Furthermore, it is increasingly important that any penalty in operating efficiency incurred by noise reduction methods be minimized.

In general, noise reduction can be achieved in two ways: (1) reduction through the attenuation of propagating sound fields; and (2) reduction of the strength of the noise sources themselves. The first approach typically involves the use of absorptive duct liners and splitters, and possibly basic modifications to the inlet duct geometry. Because add-on features are required, and the duct length may be increased, the penalties associated with this approach are added weight and some direct reduction in aerodynamic efficiency. Furthermore, there may be a degree of noise generation associated with some treatment modifications, such as in-duct splitters, particularly if the inflow to the fan is disturbed.

The second approach, which is the reduction of noise at the source, can be pursued in many ways. The basic fan design parameters can be chosen to give more favorable acoustic behavior. For instance, the tip speed can be reduced, the spacing between the rotor and stator can be increased, and the number of blades and vanes can be altered to prevent the propagation of certain duct modes. Whether these options can be exercised in a given case depends on the design constraints on the performance and size of the system.

Because, in most circumstances, acoustic considerations cannot dictate the choice of basic fan design parameters, other means of noise source reduction are worthy of consideration. These other means of source reduction necessarily involve changes in the aerodynamic design of the blades and vanes. The design changes may occur either within the framework of conventional design practice, such as the use of optimized blade section properties, or may involve the exploration of novel concepts. Although development of all the design data needed for implementing novel concepts for noise source reduction may be initially difficult, the noise reduction potential of a successful concept may greatly exceed the reduction obtained by more conventional means. Of course, the final test of an acoustically successful concept must always be whether any associated penalties in performance, complexity, and system integration can be overcome or, at least, justified in relation to the benefits.

The subsonic leading edge rotor is implemented by tailoring (sweeping) the rotor leading edge to the mean inflow such that subsonic Mach number flow is achieved normal to the leading edge along the entire span, thus preventing shock generation. Previous use of partially-swept transonic rotors was done in an effort to reduce transonic drag rise and thus improve stage efficiency. Swept stators have been previously used to reduce noise, but the design concept implemented here involves tailoring the leading edge shape to a detailed estimate of the rotor wake field incident upon the stator.

The remainder of this report is organized to describe in detail the rationale for selection of the particular concepts (Sections 2 and 3), details of the design procedure used on the swept rotor blades (Section 5) and stator vanes (Section 6), residual noise sources (Section 7), and facility integration (Section 8). Appendices contain a listing of aerothermodynamic design parameters (App. A), a discussion of geometric considerations for subsonic leading edge rotor blades (App. B), a detailed discussion of acoustical considerations in the stator design (App. C), discussion of empirical estimates of fan noise levels (App. D), and a useful algorithm for estimating trace speeds of rotor wakes on stator vane leading edges (App. E).

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SECTION 2

TRANSONIC FAN NOISE SOURCES

This section summarizes the major noise sources and mechanisms encounted with transonic fans. Typical design characteristics of single stage transonic fans are summarized in Table 1.

TABLE 1. TYPICAL CHARACTERISTICS OF SINGLE STAGE TRANSONIC FANS.

Pressure Ratio Range1.4 - 1.8Tip Speed300 to 600 m/s (1000-2000 ft/sec)Relative Rotor Tip Mach No.1.1 - 1.8Rotor Inlet Hub/Tip Ratio.35 - .50Stator Hub Mach No..8

The most important noise sources, which involve both the rotor and stator, are:

Shockwave noise from the supersonic portion of the rotor blades, often called multiple pure tone (MPT) noise.

Rotor/stator interaction noise caused by unsteady loading due to aerodynamic interaction (tonal and broadband noise).

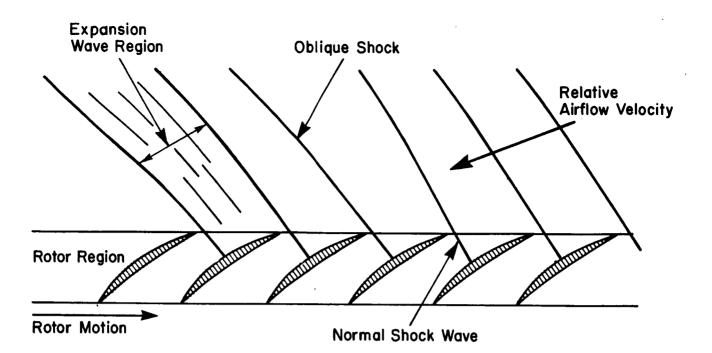
Noise caused by unsteady loading on rotor blades interacting with inflow distortions and turbulence (tonal and broadband noise).

A brief elaboration on each of these sources is now provided.

2.1 Shockwave Noise

When the relative flow past the rotor becomes supersonic, the propagation of shock waves out of the inlet duct becomes an important noise source. The upstream propagation of waves from blade rows with detached and attached shock wave patterns is shown in Fig. 1 (from Ref. 1). Because the pressure field must satisfy a periodicity condition, expansion waves occur in the regions between the shock waves.

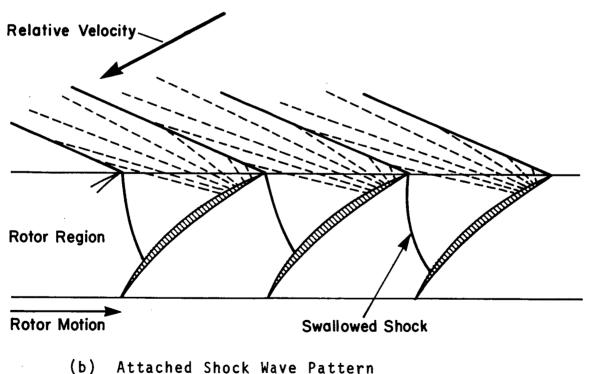
Several investigations (Refs. 2 through 7) have shown that nonlinear effects are an important factor in the upstream shock propagation process. Because nonlinear attenuation occurs more rapidly for higher initial levels, an increase or reduction of the



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(a) Detached Shock Wave Pattern



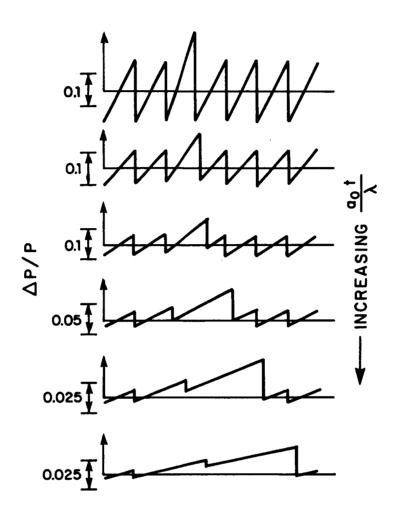
- Attached Shock Wave Pattern
- POSSIBLE SHOCK WAVE CONFIGURATIONS FOR ROTORS IN SUPERSONIC FLOW. FIG. 1.

shock strength at the blades does not produce a comparable increase or reduction of levels at the end of the inlet duct, or in the far field. This effect is strongest when the wave train in the duct is well ordered and can be considered nearly one-dimensional in character. The important consequence of this effect is that very substantial levels of source reduction must be achieved to guarantee a worthwhile reduction in level in the far field.

Another important consequence of nonlinear propagation is the redistribution of the shock noise spectrum from blade passage frequency and its harmonics to the rotor shaft rotation frequency and its harmonics. This redistribution occurs because of blade-to-blade differences in the initial strength and position of the shock waves. These blade-to-blade differences are caused by variations in manufacturing tolerances that may affect the circumferential location, setting angle, thickness, and camber of the blades. Because the shock train structure is inherently unstable to perturbations in strength and position, these initial disturbances need not be large. As an example, when periodic variations in shock strength occur, the stronger shocks tend to overtake and dominate the weaker shocks because of nonlinear effects. Because the variations in strength are caused by blade-to-blade differences, they are periodic in the shaft rotation speed. Thus, as the wave train propagates, the harmonics of shaft speed become increasingly important relative to the harmonics of blade passage frequency. Fig. 2 shows the redistribution of energy from blade passage frequency to shaft rotation frequency as the result of an initial amplitude perturbation to one shock in a wave train. Figure 3 shows sketches of typical noise spectra for a subsonic fan, which has no shock noise, and for a supersonic fan, where the tones at the harmonics of shaft speed are clearly present. Clearly the multiple pure tone noise due to shock wave propagation is a major noise problem.

2.2 Rotor/Stator Interaction Noise

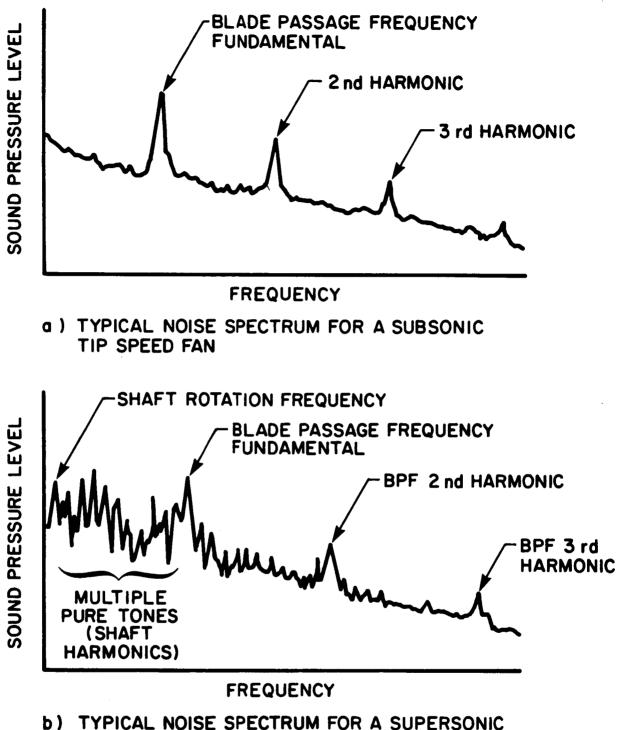
Unsteady aerodynamic loads on rotor blades or stator vanes produced by the aerodynamic interaction between the rotor and stator are an important source of both tonal and broadband noise. The main causes of the aerodynamic interaction are the interference with the potential flow pressure and velocity fields and the interaction with the viscous and turbulent wakes from upstream blades. The potential field interaction that produces tonal noise at the harmonics of the blade passage frequency can be virtually



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FIG. 2. DEVELOPMENT OF A SHOCK TRAIN WITH AN INITIAL DISTURBANCE (from Ref. 5).



- TIP SPEED FAN
- FIG. 3. TYPICAL FAN NOISE SPECTRA FOR SUBSONIC AND SUPERSONIC TIP SPEEDS.

eliminated by providing adequate spacing between the rotor and stator. Increasing the spacing on a high by-pass ratio fan stage is usually practical and does not involve a severe aerodynamic penalty. The interaction of the stator vanes with the "mean component"(steady velocity deficit) of the rotor wakes produces tonal noise at the harmonics of blade passage frequency, while the interaction with the wake turbulence produces broadband noise. Increasing the spacing between the rotor and stator also reduces - but does not necessarily eliminate - this noise source.

2.3 Inflow Distortion Noise

The inflow to the fan rotor typically exhibits a degree of spatial nonuniformity and a certain amount of turbulence. Sound is produced by unsteady loads on the rotor blades operating in this disturbed inflow. Steady spatial nonuniformity causes tonal noise to be produced at the harmonics of blade passage frequency, and the presence of turbulence produces broadband noise. However, if the turbulence scales are sufficiently long in the streamwise direction, then many blades will interact with a given disturbance in a similar manner, producing peaks in the noise spectrum at the harmonics of blade passage frequency. Because the basis for this noise source is a random process, the amplitude of these peaks will vary in time in a random manner. Inflow distortions have been shown to be a potentially important noise source in static fan test facilities. Their importance in an actual flight environment is less certain, since the effect of forward motion is usually to reduce certain types of inflow distortion.

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SECTION 3

NOISE SOURCE REDUCTION CONCEPTS

In this section, the concepts for the reduction of rotor and stator noise sources are described. A review of the detailed analysis and design procedures associated with the implementation of these concepts in the present program is postponed to the sections later in the report dealing with detailed design.

3.1 Rotor Noise Reduction

As discussed in the previous section, two noise sources associated with the rotor are multiple pure tone noise due to shock waves and inflow distortion noise. This section describes a concept which has the potential to substantially reduce multiple pure tone noise. As an additional advantage, this concept will also help reduce the problem of inflow distortion noise.

In principle, upstream-propagating shockwave noise can be reduced by designing for careful alignment of the relative velocity, w, with the suction surface near the rotor blade leading edge, as shown in Fig. 4a. However, completely shockfree entry into the blade row cannot be achieved in conventional blading because of the finite thickness of the blade leading edge. The effect of thickness is illustrated in Fig. 4b. Moreover, since the relative inflow direction varies with the operating conditions, the proper alignment cannot be maintained in off-design operation, nor in the presence of inflow distortions. Thus, this concept presents several practical difficulties for application to aircraft fans which do not operate at a single design point.

A different approach to obtain shockfree entry into a blade row is now described. It is believed that this approach does not suffer from the shortcoming of the more conventional approach just described. Consider a blade whose leading edge is swept relative to the local inflow velocity vector. The leading edge would in general appear swept when viewed from the side and skewed when viewed from the front. If the leading edge is swept such that the Mach number of the relative flow component normal to the leading edge is everywhere subsonic, a shockless leading edge results. In wing theory, this is referred to as a "subsonic leading edge in supersonic flow" (See, for instance, Ref. 8). In rotating applications, the radial variation in relative Mach number makes it possible. in principle, to completely avoid upstream shock wave propagation by using leading edge and surface generating line sweep which varies from hub to tip. In practice, structural constrains force some design compromises. In the present design, the structural

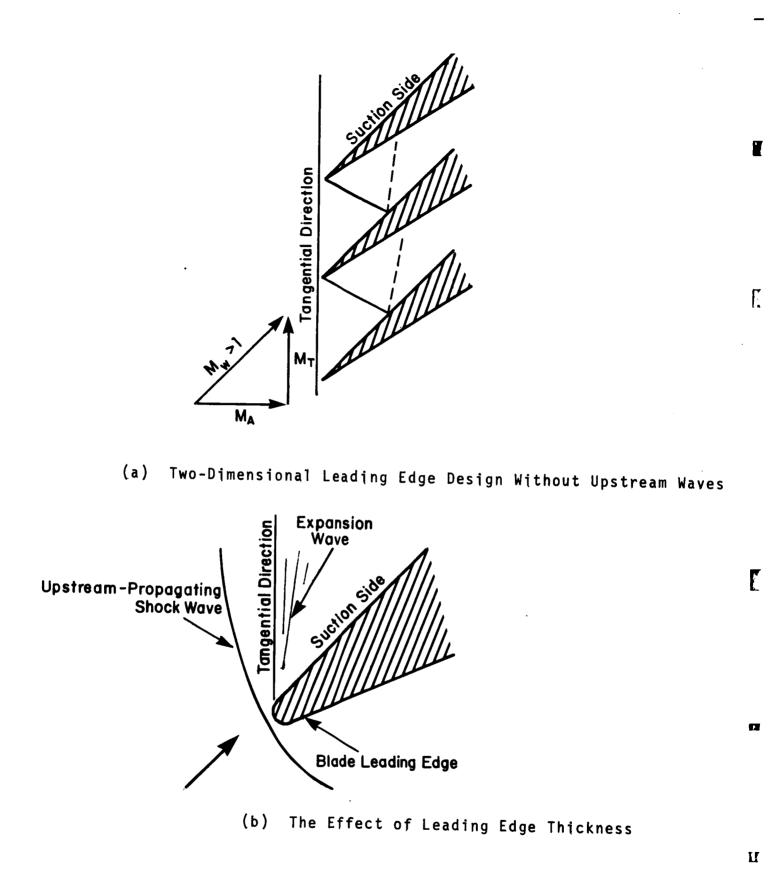


FIG. 4. SHOCKLESS LEADING EDGE DESIGN AND THE EFFECT OF THICKNESS.

compromise entails the presence of a train of conical shocks upstream of the rotor associated with a sweep discontinuity in the leading edge. From the standpoint of preventing shock noise, the design can be made insensitive to operating conditions, relative flow alignment, and inlet distortions by designing the sweep distribution for the highest relative inflow Mach number to be expected; thus ensuring a lower subsonic normal Mach number component for off-design conditions. This insensitivity is considered to be a major asset.

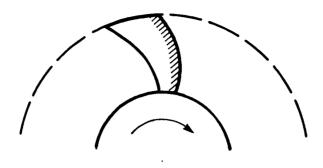
The underlying aerodynamic idea is now reviewed. Figure 5a shows a swept wing of infinite extent subject to an incident supersonic flow. Since there is no spanwise variation in the wing geometry, the axial component has no effect. The aerodynamic forces are determined entirely by the component of the flow normal to the wingspan. If the component normal to the span is subsonic, then there are no shock waves associated with the flow over this wing. Of course, to be completely shockless, the normal component must be sufficiently subsonic that transonic flow effects do not occur in the normal flow plane. The only effect of the axial component is in the structure of the viscous boundary layer on the wing surface, but this is not related to the presence or absence of shock waves. The same ideas are applicable, of course, to an infinite span sweptback cascade. Fig. 5b shows a finite span wing sweptback to have subsonic leading edges. The aerodynamic behavior is now considerably more complicated. In particular, the presence of conical shocks at the front and rear of the wing root and at the rear of the tips is unavoidable. These isolated points on the wing are discontinuities in the otherwise subsonic edges. The conical shocks are, however, weaker than their twodimensional counterparts and, because of their three-dimensional nature, decrease in strength with distance from their point of origin.

The application of a subsonic leading edge to a fan blade is illustrated in Fig. 5c. This illustration is simplified to its essential form, showing only the radial change in Mach number. The actual process is nonplanar because of the change in direction of the inflow with radial location. The particular case illustrated applies to a transonic fan, since part of the incident flow is subsonic. Then the leading edge can be made completely shockless even though the blade is of finite extent assuming that one is able to predict and accommodate the effects of spanwise flows (Ref. 9).

The local leading edge sweep at each radial station is chosen to be greater than the Mach angle of the local flow, i.e., the swept edge must lie within the local Mach cone. This assumes that the normal flow to the leading edge is everywhere subsonic. Because of the gradient in Mach number, the incident flow is subsonic at the base of the blade so a shock cannot emanate from this point (unlike the wing root in Fig. 3b). Hence, the blade leading edge can be entirely shockless, except for the effects of aerodynamic interference between the blade tip and the shroud which produce conical shocks. If the fan were completely supersonic, a conical shock should also occur at the root of the blade. By designing the leading edge and the other generating lines of the forward portion of the blade surface to be subsonic for the situation that produces maximum relative flow Mach number, the edge will remain subsonic under all other operating conditions. The blade leading edge would usually be designed to have a constant normal velocity (Mach number) component at all points along the span at radii (from the hub) greater than that at which the critical normal Mach number, M_{wcrit} , is reached. The critical Mach number is that normal Mach number (<1) at which thickness effects would cause the flow to become transonic.

In addition to sweepback, Figs. 6a, 6b, and 6c show swept forward and compound sweep blades that are also possible configurations. All of the blade configurations must have a conical shock at the tips caused by aerodynamic interference with the shroud. The compound sweep blade will also have a weak conical shock at the discontinuity in sweep, which is positioned somewhere along the leading edge (assuming the discontinuity lies in the region of supersonic relative inflow). Although the compound sweep blade has the acoustic penalty of introducing a weak conical shock, it offers other definite advantages. Structural considerations provide the most severe constraint to the design of high speed fans with swept blades. Fairly large excursions of the leading edge are required to implement this concept. It should be noted that the family of threedimensional curves that satisfies the subsonic leading edge condition is not unique and therefore considerable latitude exists to determine structurally optimum shapes. Figure 7 shows the type of conical shock wave pattern for a compound sweep blade. The blade in the sketch closely resembles the design developed during the course of the project being described.

Figure 8 compares the operation of a moderately loaded blade row with and without subsonic leading edges in supersonic flow. As explained above, the subsonic edge region allows shock-free entry into the blade row. The blade rows are identical except for the addition of a subsonic leading edge region in one case. The front surface of the blade must be designed so that any shocks generated on the suction surface of



A. Swept Back

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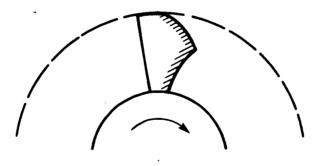
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B. Swept Forward



C. Compound Sweep

FIG. 6. FRONT VIEW OF SOME POSSIBLE BLADE CONFIGURATIONS WITH SUBSONIC LEADING EDGES.

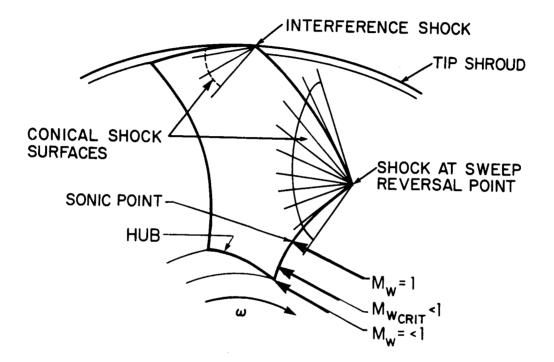
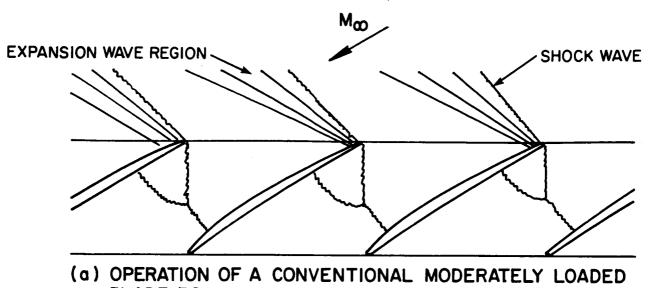
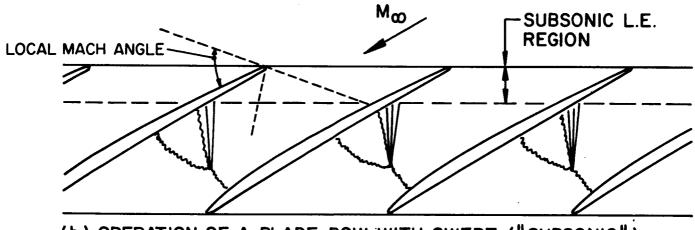


FIG. 7. CONICAL SHOCK FIELD FROM A ROTOR BLADE WITH A COMPOUND SWEEP LEADING EDGE.



BLADE ROW



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(b) OPERATION OF A BLADE ROW WITH SWEPT ("SUBSONIC") LEADING EDGES

FIG. 8. COMPARISON OF THE OPERATION OF A MODERATELY LOADED BLADE ROW WITH AND WITHOUT SUBSONIC LEADING EDGES. the blade are formed sufficiently far back that the disturbance is entirely contained in the blade row, even during off-design operation.

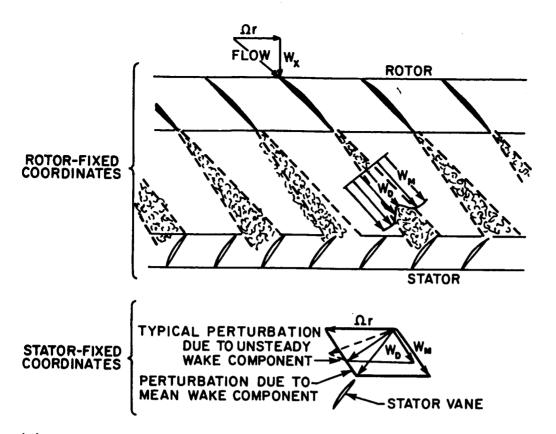
Using a swept leading edge also helps reduce the response of the rotor to inflow distortions, because the magnitude of the response is largely determined by the velocity component normal to the leading edge (Ref. 10). The effect of inflow distortion is most important near the tip of the rotor where the relative velocity is highest. Fortunately, the concept for sweeping the blades requires the most sweep near the tip.

3.2 Stator Noise Reduction by Leading Edge Sweeping and Blade/ Vane Number Selection

Although increasing the spacing between the rotor and stator leads to some noise reduction, the aerodynamic interaction between the rotor wakes and stator vanes remains an important noise source. Further reduction by conventional means can be achieved by choosing the proper number of blades and vanes to cut off many of the acoustic spinning modes in the duct (Ref.11). When the rotor tip speed is subsonic, the blade and vane numbers can be chosen so that all the spinning modes at blade passage frequency, and at least some of the modes at higher harmonics, are cut off. However, if the rotor tip speed is supersonic, at least one spinning mode at blade passage frequency cannot be cut off, regardless of the choice of blade and vane numbers. Since supersonic spinning speeds often occur on transonic fan designs, other means of stator noise reduction are of considerable interest.

Figure 9a illustrates the interaction of a row of stator vanes with rotor wakes when viewed on a surface of constant radius from the fan axis. The wakes can be described as flow regions with an average velocity \overline{W} lower than the velocity of the adjacent fluid, upon which a turbulent perturbation velocity field Δw is superimposed.

Figure 9b shows a sketch of a three-dimensional wake/vane interaction in a fan. The structure of the viscous, usually turbulent, wakes that trail each rotor blade is complex. However, on the average, these wakes can be considered as being convected with the mean flow in which they are imbedded. The nature of the downstream mean flow is such that the convection process will distort the wakes from their original shape; namely, the downstream flow is distorted both axially and circumferentially across a given radial path, leaving the downstream pattern of the wake disturbance very much altered from the pattern at the rotor trailing edge. Suppose, for instance, the rotor is designed to give a mean flow that has a uniform axial velocity distribution and a free vortex tangential velocity distribution. Assuming

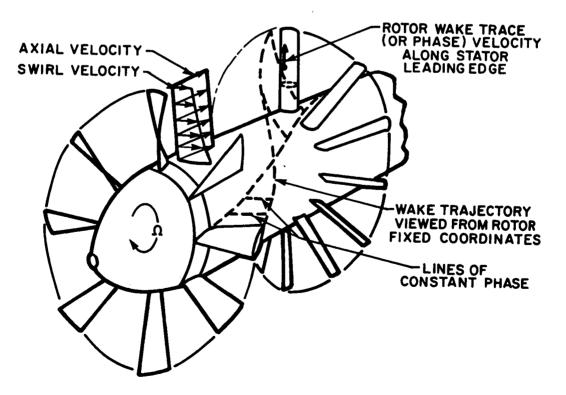


(a) The Interaction of the Stator Vane Row with the Mean and Unsteady Rotor Wake Components as Seen on a Constant Radius Surface. I

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- (b) A Sketch Showing the Three-Dimensional Nature of the Rotor-Wake/ Stator-Vane Interaction.
 - 18 FIG. 9. THE CHARACTERISTICS OF ROTOR-WAKE/STATOR-VANE INTERACTION.

the wakes are radial at the rotor trailing edge, it is clear that the tangential velocity component will act to skew the wakes over, with the hub region leading the tip region. This situation is illustrated in Fig. 9b. In this case, the interaction of a given wake with a given stator vane does not occur simultaneously all along the stator vane span. Instead, the instantaneous spanwise interaction region of a single rotor wake will extend over only a portion of any one vane and will sweep along the vane leading edge, beginning at the hub and ending at the tip. The skewing of a wake due to convection by the downstream mean flow can be sufficient to involve simultaneously portions of several stator vanes.

The shape of wake and the magnitude of its velocity components vary from hub to tip. To complete this picture of the downstream flow field, one must consider the unsteady velocity components which account for the turbulent structure of the wakes and for any other sources of inhomogenieties in the flow, e.g., inlet flow distortions, large-scale flow instabilities, and blading errors. In general, the statistical properties of these unsteady components can be expected to vary axially, circumferentially, and radially.

Both the mean and unsteady velocity components of the wake flow induce unsteady loads on the stator vanes. The mean component will produce a load distribution that travels from hub to tip, changing shape and amplitude in accordance with the radial variation of the mean flow properties and wake strength, width, and skew. Imposed on this traveling load distribution will be the unsteady effect of the turbulent structure of the wake. The end result of all sources of unsteady loading on the stator vanes is to produce tonal and broadband noise. The tonal noise is usually considered to be the more important noise source. The speed at which the point of interaction of the flow disturbance with the vane travels along the span is called the trace speed.

A particular source of unsteady loading will produce no significant acoustic radiation if it satisfies a subsonic and non-accelerating trace speed criterion along the vane span. The trace speed concept has been previously recognized for the problem of helicopter-blade/vortex interaction by Widnall (Ref. 12) although it has not been generally recognized in the study of fan noise.

The interaction of the wake with the vane produces a load distribution that travels along the vane. Suppose the vane is much longer than an acoustic wavelength. Following the trace of a phase front of this load distribution, acoustic radiation can occur along the vane span if the magnitude of the load changes,

the phase speed changes with time, or if the phase speed is supersonic. For instance, in fan noise analyses the rotor-wake/ stator-vane interaction is usually assumed to be two-dimensional (corresponding to infinite spanwise trace speed). The conditions mentioned above are necessary for radiation but not sufficient. The interaction with the acoustic field produced by the other vanes must also be considered before the actual occurrence of acoustic radiation can be established. Therefore, regions along the stator vane span can be expected to be poor radiators if the phase speeds are subsonic, nearly constant, and local levels do not vary rapidly. Other regions may or may not be efficient radiators depending on the behavior of the distribution of sources elsewhere on the stator. Furthermore, end effects at the hub and tip (within approximately one half an acoustic wavelength of the ends) makes these regions potential radiators. These considerations are discussed in Appendix C, and justified in detail in Bliss, et al., (Ref. 13).

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Understanding the rotor-wake/stator-vane interaction and the criteria for radiation from the span of a single vane suggests ways in which the vane configuration can be altered to achieve noise reduction. The vane should be shaped so that loads traveling along the span move at a constant subsonic speed. Assuming that the amplitude of the load distribution. moving with a phase front, is essentially constant, then radiation from the vane span will not occur (except for endeffects). The condition of a constant subsonic spanwise trace speed can be achieved by sweeping or skewing the stator vanes, as illustrated in Fig. 10. In this illustration, the lines of constant phase can be considered to be the intersection of the rotor wakes with the plane of observation (e.g., the r- θ plane in Fig. 10a, and the r-z plane in Fig. 10b). Except for the effect of shape changes, these lines travel at constant speed (rotational in the $r-\theta$ plane and rectilinear in the r-zplane) because of the rotation of the rotor. The speed at which a phase front traces the leading edge of the stator vane depends on the shape of the leading edge and the shape of the phase front. Clearly the trace speed can be controlled by either sweeping or skewing the stator vane. With this approach, radiation from the stator span can be prevented, leaving only acoustic radiation from end effects at the hub and tip of the vane. Radiation from the hub region can be cut off by the proper choice of blade and vane numbers, provided that the rotation speed of wakes at the hub is subsonic. Since the rotation speed of wakes at the stator tip will usually be supersonic for a transonic fan, the radiation from tip end effects can never be entirely cut off. Note that the rotor

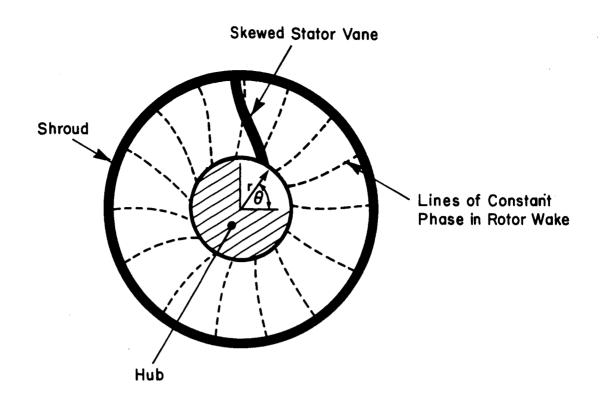
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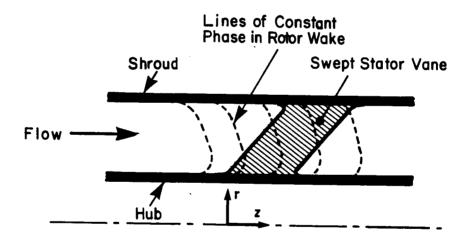
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(a) Schematic Axial View (r- θ plane)-Skewed Vane



(b) Longitudinal Section (r-z Plane)-Swept Vane

FIG. 10. SCHEMATIC OF PHASE SHIFTS BETWEEN ROTOR WAKES AND SKEWED/SWEPT STATOR VANE.

wake pattern rotates with the same angular velocity Ω as the rotor. Thus, at any given radius at any downstream location between the rotor and stator, the rotation speed of the wake pattern is simply, Ωr , which is different than the swirl velocity component. This can be best visualized from rotor fixed co-ordinates from which the wake pattern appears "frozen."

Another, but related, way to view the effect of sweeping or skewing the stator vanes is as follows. Tyler and Sofrin (1962) have shown that for a given circumferential mode number, m, and hub-to-tip ratio, v, the radial structure of an acoustic spinning mode can be described by an infinite series of characteristic functions. The functions in this series differ according to their radial order, μ , i.e., each function has a different number of nodes in the interval between the hub and tip. The spinning speed at which each of these functions begins to radiate is always supersonic and increases with increasing radial order. Therefore, at a given supersonic spinning speed and fixed m and σ , only a certain number of the functions corresponding to the lowest radial order will not be cut off. Vanes can be skewed or swept so that the number of wakes on a given vane is increased, raising the radial order of the load distribution on the vanes. The acoustic energy is thereby redistributed to higher radial orders, some of which will be cut off. The relationship between duct mode cut off and the constant subsonic trace criterion is discussed by Bliss, et al., (Ref. 13), and in Appendix C.

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SECTION 4

FAN STAGE DESIGN SUMMARY

An experimental transonic fan stage was designed and constructed using the noise reduction concepts explained in the two preceding sections. The fan uses compound sweep rotor blades designed to have "subsonic leading edges" in the region of supersonic relative inflow. The stator vanes were swept back to achieve a constant subsonic trace speed of rotor wakes along the vane span. Figures 11a, b and c show photographs of the actual fan stage. A cross-sectional view of the fan as it will appear when installed in the test facility of NASA Lewis is shown in Fig. 12. As indicated in the illustration, the fan will be tested in both forward and reverse installation arrangements in order to measure both the fore and aft noise characteristics. The design data for the fan stage is summarized in Table 2. In the remainder of the report, the detailed design procedures used in the development of the fan stage are described.

TABLE 2. FAN STAGE DESIGN SUMMARY

Stage Characteristics:

Stage Pressure Ratio, $P_4/P_1 = 1.6$ Mass Flow Rate, W = 31.2 kg/s (68.8 lb/sec) Specific Mass Flow Rate:(referred to annular area at rotor inlet) $W_{as} = 199.03$ kg/s·m² (40.76 lb/sec·ft²)

Polytropic Stage Efficiency, n = 0.86

Rotor:

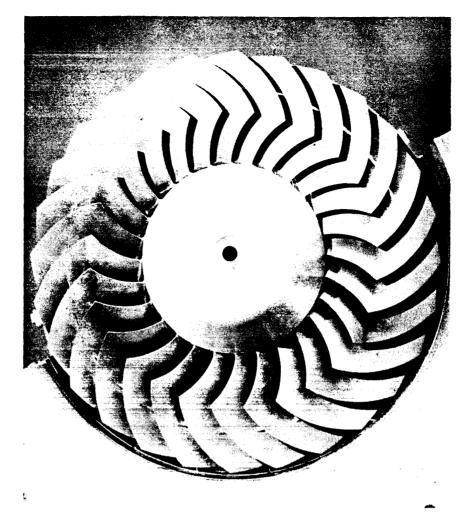
28 Compound Sweep Blades Leading Edge Normal Mach Number = 0.91 Tip Speed = 480 m/s (1575 ft/sec) Relative Tip Inlet Mach Number = 1.593 Rotor Inlet Tip Radius = 249 mm (9.803 in) Rotor Inlet Hub-Tip Ratio = 0.442 Rotor Pressure Ratio, P_2/P_1 =1.64

Stator:

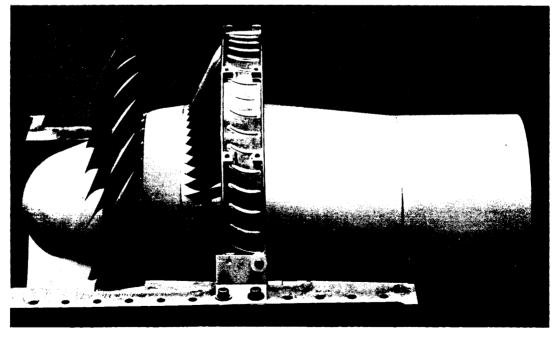
59 Swept Back Vanes Sweep Angle = 25° At Root, 40° At Tip Stator Inlet Mach Number = 0.80 Stator Pressure Loss $\Delta P_{3-4}/P_3 = .025$



(a) 45° VIEW OF ROTOR AND STATOR ASSEMBLY FROM FRONT OF FAN FIG. 11. PHOTOGRAPHS OF THE EXPERIMENTAL FAN STAGE

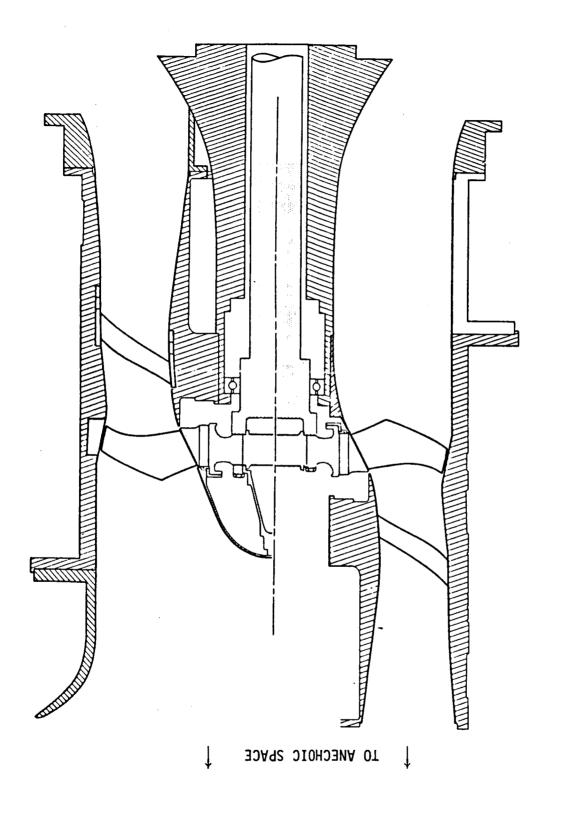


(b) Front View



(c) Side View

FIG. 11 concluded PHOTOGRAPHS OF THE EXPERIMENTAL FAN STAGE.



FORWARD AND REVERSE INSTALLATION IN THE W2 FAN NOISE TEST FACILITY AT NASA LEWIS. FIG. 12.

SECTION 5

DETAILED ROTOR DESIGN

This section deals with all aspects of the detailed analysis and design procedures associated with the fan rotor. The aerodynamic design of a transonic rotor having blades with "subsonic" leading edges differs significantly from conventional design practice because the acoustic, aerodynamic, and structural requirements interact strongly with each other from the very beginning of the preliminary design phase. Therefore it was necessary to conduct numerous aerodynamic-acousticstructural design iterations to optimize and finalize a rotor configuration satisfying all the design requirements.

The overall design point data for the rotor are listed in Table 2 of the previous section.

5.1 Aerodynamic Design

Certain differences are to be expected in the aerodynamic behavior of a rotor with subsonic leading edges. Since the entry into the blade row is shock free, any shocks that occur must remain within the blade row under all operating conditions because the edge region cannot support a shock system. Furthermore, the effects of sweep may introduce other three-dimensional flow phenomena which are not present in a conventional blade design. Given these facts, the rotor aerodynamic design was undertaken using the best conventional design practice combined with an anticipation of the most important effects of swept edges. The design was carried out primarily with the use of an axisymmetric flow computer program. Conventional (twodimensional) methods for analyzing the flow behavior within the blade row are not really adequate for the three-dimensional case of blades with swept "subsonic" edges. To handle this problem analytically requires a more general approach. Some work was done to adapt a new fully three-dimensional computer code to the analysis of flow through the blades with swept edges, but was discontinued due to schedule requirements.

An important question in the design of a rotor with "subsonic" edges is related to its surge margin. Because the edge region cannot support a shock system it was felt that the surge margin might be reduced. Such a reduction would occur, if the effective rotor operating range were limited by the condition that the shock system remain within the covered cascade region. The flow configuration in which the shock system must remain within the covered cascade, however, does not yield the maximum static pressure rise achievable in a conventional transonic-supersonic rotor. Consequently, in a stage where surge is not triggered prematurely by the stator flow conditions, a rotor with subsonic leading edges might result in a decrease of the surge margin as compared to a conventional design.

Since the rotor aerodynamic loading essentially depends upon the rotor static pressure rise, the selection of the meridional flow path was the main design step taken to achieve the desired loading levels. Meridional channel conicity and curvature through the rotor section were traded off in several preliminary design attempts. The flow calculations were performed by means of a code which solves the general equation of radial equilibrium on straight axial or slanted stations for the axisymmetric flow case taking into account the radial variation of the blade efficiency. The polytropic efficiency η assumed for the rotor blading is shown in Fig. 13, where η is derived from

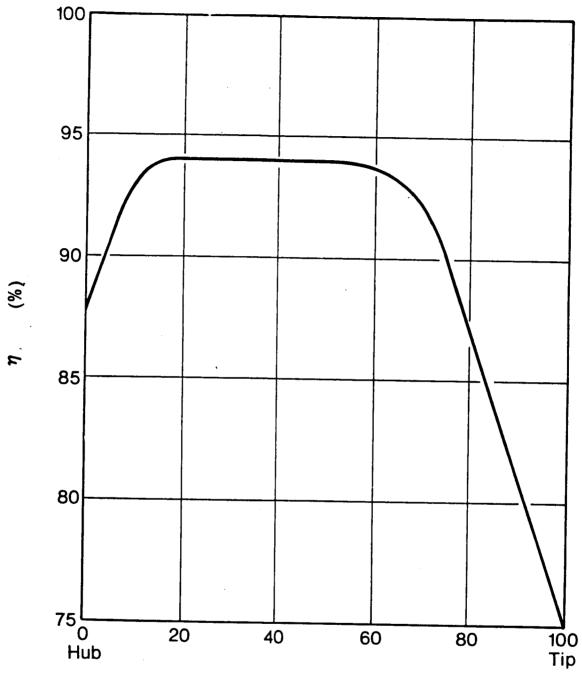
 $\frac{P_{2}}{P_{1}} = \left(\frac{T_{2}}{T_{1}}\right)^{(\eta) \left(\frac{\gamma}{\gamma-1}\right)}$

It was found that the comparatively large channel conicity across the rotor section required by the high design pressure ratio $P_4/P_1 = 1.6$ and the wall curvature needed at rotor exit to prevent excessive channel contraction in the free space between rotor and stator, combine to shift the maximum rotor static pressure rise from the tip towards the midspan location, where the shock system has the greatest tendency to move upstream into the uncovered cascade region because of the lower relative inlet Mach number. The main preliminary design effort consequently was directed toward minimizing the static pressure rise at the critical midspan location.

The optimum channel configuration is shown in Fig. 14. The flow conditions are summarized on the Aero design program (R-121) input and output printout attached in Appendix A.

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Figure 15a shows the distribution of the rotor static pressure ratio P_2/P_1 over the channel height, together with the relative inlet Mach number M and the corresponding normal shock pressure ratio $\hat{P_1}/P_1$, which is roughly equivalent to the static pressure ratio obtained in the front portion of a conventional cascade with a normal shock attached to the leading edge.



Percent Blade Span

FIG. 13. ROTOR POLYTROPIC EFFICIENCY η

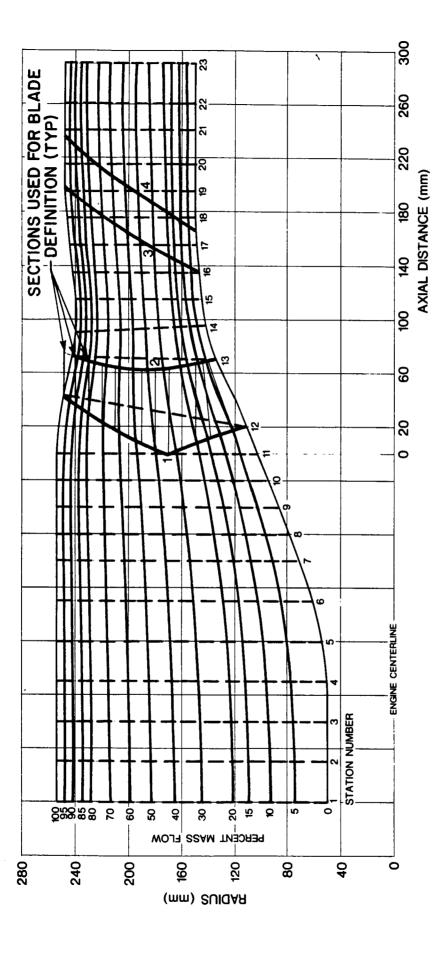


FIG. 14. MERIDIONAL FLOW PATH.

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Three types of operating conditions can be distinguished along the rotor blade span.

(a) From the hub to the sonic radius $r_1(M_{w_1} = 1)$, the rotor

static pressure rise is achieved essentially by subsonic relative flow deceleration and centrifugation.

(b) From
$$r_1(M_w = 1)$$
, to $r_1(P_2/P_1 = \hat{P}_1/P_1)$ the rotor static

pressure rise must be achieved through a channel-contained normal shock or a pseudoshock system followed by subsonic relative flow deceleration. The radial distribution of the rotor static pressure ratio P,/P, determined how far this operating condition extends beyond the sonic radius. In the present case, it extends roughly from the 12% to the 40% mass flow streamline, or from 20% to 53% of the span, i.e., slightly beyond the point of sweep reversal. The maximum inlet Mach number in this blade section remains below the 1.3 level at which the interaction of normal shock with a turbulent boundary layer produces extensive flow separation ($\hat{P}/P = 1.8$). If minor flow separation does occur in the upper portion of this region, the flow will reattach to the blade because of the large solidity provided in the vicinity of the point of sweep reversal. Consequently, it is expected that the design flow conditions will be obtained over this critical span section by a shock configuration located in the forward, yet still covered portion, of the cascade.

(c) In the upper blade section, P_2/P_1 is smaller than P_1/P_1 , and the shock system consequently moves progressively toward the rear portion of the cascade. Since no shock is attached to the leading edge, the flow conditions are essentially similar to those in the diverging section of a converging-diverging nozzle in the supersonic off-design operating range.

Figure 15b schematically shows the meridional projection of the rotor blade and the anticipated shock/pseudoshock interception area on its pressure and suction sides. The main question pertains to the rotor surge margin, i.e. the extent to which the tip region will be allowed to increase its pressure ratio beyond the design value by forward shifting of the shock configuration before (i) flow separation occurs at the hub, or (ii) the shock system at midspan is forced into the uncovered cascade region.

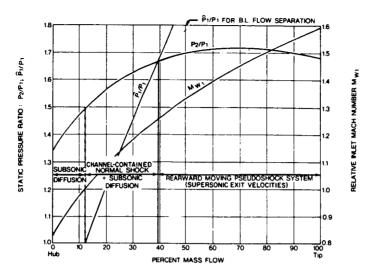


FIG. 15(a). ROTOR FLOW CONDITIONS: SPANWISE DISTRIBUTION OF STATIC PRESSURE RATIO AND INLET RELATIVE MACH NUMBER (Design Point).

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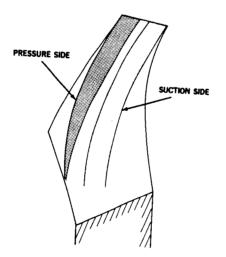


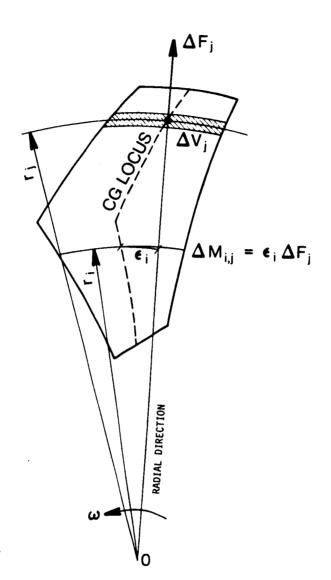
FIG. 15(b). ROTOR BLADE SHOCK/PSEUDOSHOCK INTERCEPTION AREA (Schematic).

5.2 Description of the Aero-Structural Design Interaction Problem

The problem of achieving acceptable stress levels is much more difficult for a rotor with swept leading edges than for a conventional design. The aerodynamic and structural requirements for the rotor blade are therefore closely coupled. Within the aerodynamic constraints, a number of design iterations were required to achieve acceptable stress levels and to optimize the The major aerodynamic constraints are that the rotor design. meet the design performance requirements and that the normal component of flow to the leading edge be maintained at a certain subsonic value. Because of the gradient of relative inflow Mach number, the angle of sweep must increase toward the tip to meet the condition of a subsonic normal component. In the present case, the maximum normal component Mach number was chosen to be 0.91 along these leading edges (actually lower near the hub). The value of 0.91 was chosen as a goal since it represented a normal Mach number sufficiently below sonic to avoid thicknessrelated shocks. Lower values can be chosen, but the severity of the blade leading edge excursions increase as the normal Mach number is lowered. For the fan design tip speed, the excursions of the swept leading edge are large and it was necessary to use a compound sweep configuration to minimize bending stresses. The major variables available to control blade stresses are the location of the sweep reversal point, the local section properties of chord length, maximum thickness, thickness distribution, and the stacking of the blade sections.

Because of the large leading edge excursions, the centers of gravity of the blade sections can no longer be stacked on a radial line. In addition to the centrifugal tensile stresses, large bending moments about both principal axes of inertia of the blade sections were found to occur (Fig. 16). Achievement of acceptable stress levels required the use of a carefully chosen sweep reversal point and the development of an effective nonradial stacking procedure.

Typically, the most critical problem was the bending moments about the minor axis of inertia, and a special stacking procedure was used to minimize these moments. Α near-optimum procedure for nonradial stacking is as follows. The blade sections were stacked starting at the tip and moving inward. The addition of each incremental blade section was made so that the center of gravity of the entire portion of the blade above this section falls on the axis of minimum inertia of the new section. The center of gravity of the new upper portion was then reevaluated before the next incremental section was added in the same manner. This procedure nearly minimizes the critical bending stresses around the axis of minimum inertia. The result is not completely optimum because of the



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FIG. 16. BLADE CENTRIFUGAL FORCES AND MOMENTS.

complexity of the actual situation in which the stresses are determined by the complex interaction of many effects. Further improvements were made by iterative changes around the result of the above stacking procedure, particularly with the intention of relieving local stress concentrations. To the extent that high stresses arise due to bending around the axis of maximum inertia, these can be relieved largely by changing the location of the sweep reversal point and varying the local section chord and thickness.

5.3 Determination of the Subsonic Rotor Leading Edge Geometry

At each leading edge point, the relative Mach vector M_{w_1}

defines a Mach cone. To a prescribed value of the subsonic velocity component M $_{\rm W}$ perpendicular to the leading edge, $_{\rm W_{1}L}^{\rm W}$

there corresponds a coaxial cone with smaller aperture. The subsonic leading edge elements must only satisfy the condition that they lie on such cones. Their direction otherwise is arbitrary.

Referring to Fig. 17, a particular sweep direction can be defined by specifying that each leading edge element be swept in the plane formed by the relative inlet velocity, W, and the radius (W-r) plane. This yields the shortest leading edge line from hub to tip, since it maximizes the radial projection of every leading edge segment.

Sweeping in the W-r plane however, does not result in a blade with minimum stresses. The resulting stacking of the CG's of the profiles in fact was shown to generate substantial bending moments around their axis of minimum inertia. The main parameter used to minimize bending stresses is the lateral sweep angle, v, between the radial plane passing through the leading edge element dl and the W-r plane. The situation is illustrated in Fig. 17. The geometric analysis used for this design is described in Appendix B, and only some pertinent results are cited below. They are expressed by the two following equations for the cylindrical coordinates $\boldsymbol{\theta}_{T_{i}}$ and $\boldsymbol{z}_{T_{i}}$ of the leading edge points in function of the relative flow angle $\boldsymbol{\beta}_{,}$, the lateral sweep angle $\nu_{\text{,}}$ the slope $\boldsymbol{\epsilon}_{_{W}}$ of the relative velocity and the projection μ " of the Mach cone angle μ in the W-r plane (see Appendix B):

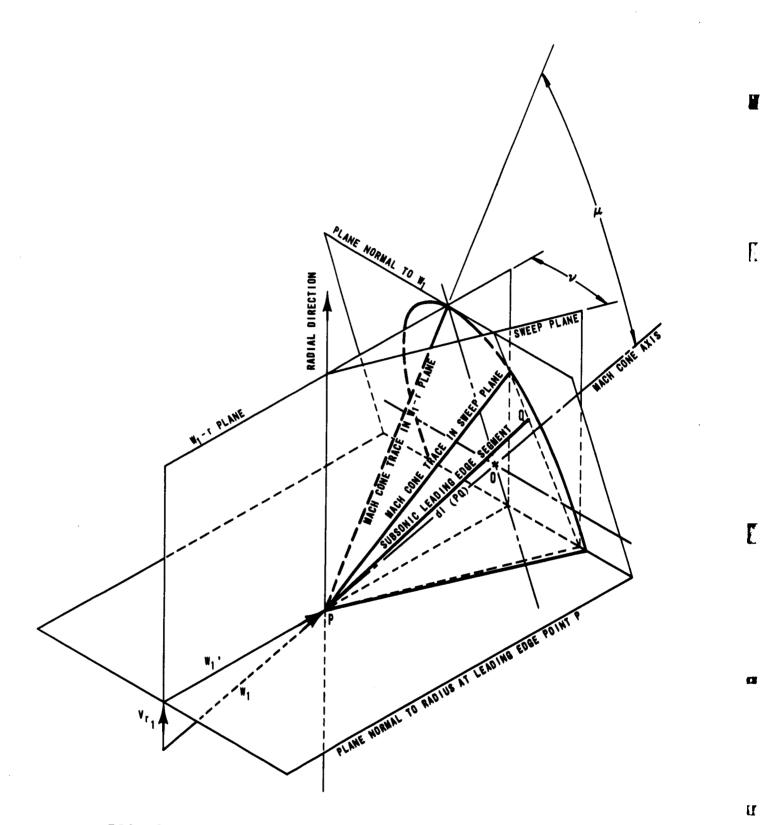


FIG. 17. SONIC SWEPT LEADING EDGE ELEMENT.

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$$\theta_{L}(\mathbf{r}) = \theta_{L_{1}} \pm \int_{\mathbf{r}_{M_{W}}=1}^{\mathbf{r}} \left[\frac{\cos(\beta_{1} + \nu)}{\tan(\mu'' \pm \varepsilon_{W_{1}})} \cdot \frac{1}{\rho \cos\nu} \right] d\rho \qquad (1)$$

$$Z_{L}(\mathbf{r}) = Z_{L_{1}} \pm \int_{W}^{\mathbf{r}} \left[\frac{\sin(\beta_{1} + \nu)}{\tan(\mu^{\prime\prime} \pm \epsilon_{W_{1}})\cos\nu} \right] d\rho$$
(2)

$$\varepsilon_{W_1} = \sin^{-1} \left(\frac{V_{r_1}}{W_1} \right)$$
(3)

The relation between
$$\mu$$
" and μ is given by the formula

$$\tan \mu'' = \frac{\pm \sin \varepsilon_{W_1} \cos \varepsilon_{W_1} \tan^2 \nu + \sqrt{\tan^2 \mu (1 + \sin^2 \varepsilon_{W_1} \tan^2 \nu) - \cos^2 \varepsilon_{W_1} \tan^2 \nu}}{1 + \sin^2 \varepsilon_{W_1} \tan^2 \nu}$$
(4)

In the above relations the (+) sign applies for backward, the (-) sign for forward sweep.

The formulae define a sonic leading edge, i.e., leading edge points lying on the Mach cones of the adjacent points. A subsonic leading edge is simply obtained by using in the formulae μ values corresponding to relative Mach numbers increased by a factor $f = 1/M_{1L}$, i.e. $M_{1}^{1} = M_{W_{1}}/M_{W_{1L}}$ where

 $\overset{M}{\mbox{W}}$ is the subsonic Mach number of the relative velocity $\overset{W}{\mbox{L}}$

component perpendicular to the leading edge. This simple relationship is illustrated in Fig. 18.

The second design parameter used to minimize the bending stresses was the sweep reversal radius. By proper selection of the point of sweep reversal, the center of gravity of the blade can be located in such as way as to project radially on, or near, the axis of maximum inertia of the hub section. From a structural viewpoint, the compound sweep blade of Fig. 6 could be considered as a blade with hub and tip sections designed and stacked according to conventional practice and fitted with an additional front section to materialize the subsonic leading edge configuration. The above CG stacking condition then could be fulfilled by similarly fitting a rear section to restore the symmetry of the mass distribution with respect to the axis of maximum inertia of the profiles. This, however, would maximize the additional blade mass and the elongation of the profile chord lengths required by the compound sweep design, which is structurally and aerodynamically undesirable. Proper selection of the radius of sweep reversal thus is necessary to ensure minimum blade stress and aerodynamic performance penalties. Adjustments of the profile chord lengths can be used only to compensate for a slightly off-optimum location of the sweep reversal point. Accordingly, the optimum stacking should yield hub stresses exceeding those of a conventional blade only by the contribution due to the blade mass added to incorporate the subsonic leading edge configuration.

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From the preliminary design iterations, the meridional projection of the subsonic leading edge line and its sweep reversal point were known with sufficient accuracy to define the radial distribution of the relative Mach numbers M_{u_1} (r) and w_{u_1} the relative flow angles β_1 (r) and ϵ_{w_1} (r) at the leading edge for final design. Those data were interpolated on the streamlines between stations 9, 10, 11, 12 of the R-121 flow calculation. (For the axial station nomenclature, refer to Fig. 14 and Appendix A.) Table 3 presents the interpolated inlet Mach numbers M_{w_1} , together with the selected Mach factors f and the corresponding Mach numbers M_w of the relative velocity component $\frac{1}{2}$ perpendicular to the leading edge and M_{w_1} of the relative velocities, introduced in Eqs. 1, 2 and 4.

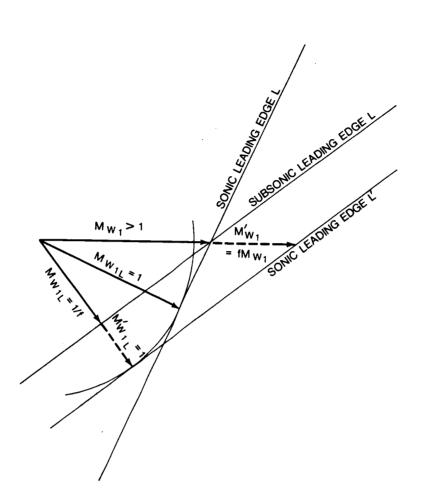


FIG. 18. SONIC AND SUBSONIC LEADING EDGES.

Leading Edge Radius (mm)	Relative Mach Nr. M _{wl}	Mach Factor f	$M'_{w_1} = f M_{w_1}$	M _{w1} L
110	.829	1.206	1.000	. 829
116	.859	1.170	1.005	.855
122	. 888	1.140	1.012	.877
129	.924	1.118	1.033	.894
136	. 959	1.107	1.062	. 903
143	.994	1.102	1.095	. 908
150	1.028	1.100	1.131	. 909
160	1.078	ł	1,186	
170	1.127		1.240	
180	1.185		1.304	
190	1.242		1.366	
200	1.302		1.432	
210	1.363		1.499	
220	1.422		1.564	
230	1.481		1.629	
240	1.539		1.693	
249	1,588	1.100	1.747	.909

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TABLE 3. Interpolated Aerodynamic Data for Final Subsonic Leading Edge Design

It will be seen that forward sweep starts immediately at the hub by setting M' = 1, i.e., by requiring that $M_{W} \equiv M_{W} = 1_{L}$.829 at the hub section. The selected values of M_{W} increase gradually to .91 at approximately 1/3 of the span in accordance with the decreasing thickness and camber of the profiles, and then remain constant up to the tip section.

The cylindrical coordinates in the lateral sweep angle ν of the subsonic leading edge line are listed in the outlined columns of Table 4, which reproduces the input/output data of the computerized calculation.

5.4 Rotor Blade Profile Definition and Stacking Procedure

The optimum profile stacking configuration can be described as follows: At every blade section along the span, the CG of the upper blade portion projects radially on or near the axis of minimum inertia of that section. This means that the radial projections of the individual CG's of the upper profiles must straddle the axis of minimum inertia of the lower section (subsequently referred to as i-straddling). This is achieved by iterative selection of the lateral sweep angle v along the span. During that iteration, the radial location of the point of sweep reversal initially selected is kept unchanged. When adequate i-straddling is obtained for all blade sections, the CG straddling with respect to the axis of maximum inertia (I-straddling) of the hub section is checked and the radial location of the point of sweep reversal modified accordingly.

The first preliminary design investigations were carried out with double circular arc profiles. In the course of the profile stacking iterations, it appeared that using airfoil sections with CG's shifting progressively backward in the lower span portion with forward leading edge sweep, and forward in the upper portion with backward sweep, i.e., a blade configuration with minimum chordwise excursion of the profile CG's, could substantially contribute to minimize bending stresses.

A simple analytical blade thickness distribution was used to simplify the design iterations involving changes in section properties to help relieve stresses. The thickness distribution is written in the following parametric form

$$t(x) = kx^{n} (c-x)$$
(5)

CYLINDRICAL COORDINATES AND LATERAL SWEEP ANGLE OF THE SUBSONIC Rotor leading edge line. TABLE 4.

<u>۵</u> .	84514L 85L47 Velacity * 4AC (4/55c) Va.
1.0300 36.400	ê
1.0151 36.000	ۍ ۲
1.0120 35.150	50
001°72 UEEU°1	J.
1.0620 33.050	č.
1,°957 32,°50	
1.1310 31.600	¢.
1.1965 29.550	15 C
	ב א ר
1.24AA 28.130	ç
1-3940 27-265	40
1.3660 26.510	
1-4320 - 25- 90C	S)
1°4997 25°350	5
1.5649 24.650	4 7
J30°£2 (627°1	C 6
1.6930 23.100	C.F.
1.7477 22.200	7.7

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where c is the chord length and n, a shape parameter. By adding a leading and trailing edge thickness,

$$t_{LE} \equiv t_{TE} = \tau v C$$

where τ is the LE and TE thickness factor and $\nu = t_{max}^{/C}$ the relative blade thickness, a practical blade thickness distribution is obtained. The abcissa for maximum thickness is given by

$$x_{t_{max}} = \frac{nc}{n+1}$$
(6)

the factor k by

$$k = \frac{\nu (1-\tau)}{\left(\frac{nc}{n+1}\right)^n \left(\frac{1}{n+1}\right)}$$

The complete non-dimensionalized formula is

$$\frac{\mathbf{t}}{\mathbf{c}} = \tau \mathbf{v} + \frac{\mathbf{v}(1-\tau)}{\left(\frac{\mathbf{n}}{\mathbf{n}+1}\right)^{\mathbf{n}}\left(\frac{1}{\mathbf{n}+1}\right)} \left(\frac{\mathbf{x}}{\mathbf{c}}\right)^{\mathbf{n}} \left(1-\frac{\mathbf{x}}{\mathbf{c}}\right)$$
(7)

For n = 1, $\binom{x}{c}_{t} = .5$. Furthermore, the second derivative is constant, so that the resulting profile is essentially a double circular arc profile for small thickness.

For
$$n > 1$$
, $(x/c)_{t max} > 1/2$ and the profile CG shifts

toward the trailing edge. Since the first and second derivatives of the thickness distribution are continuous, the profile curvature is continuous.

Using profiles with circular mean camber lines and n varying from 1 to 1.8 from the hub to the point of sweep reversal, and back to 1 at the tip section, a favorable blade configuration was obtained. However, manufacturing difficulties and the extreme sensitivity to tolerance and foreign object damage of thin profiles with n > 1.5 lead to the selection of n=1, i.e., essentially double circular arc profiles for final rotor blade design. The blade cascade geometry was defined by means of conventional procedures and criteria. Figure 19 shows representative streamline velocity triangles, together with the corresponding relative flow deceleration rates W_2/W_1 and static pressure ratios P_2/P_1 , the selected cascade solidities $\sigma = c/s$ and the resulting D-factor values. The hub and tip cascade solidities are equivalent to those which would have been selected for a conventional design with identical rotor inlet and exit flow conditions. The 30% streamline velocity triangles are representative of the conditions at the sweep reversal section (r = 170mm).

The flow deviation angles δ at rotor exit were calculated with Carter's empirical formula (Ref. 13)

$$\delta = m\phi / \sqrt{\sigma} \tag{8}$$

with m = 0.23 + 0.05 β_2 (circular mean camber line). For ax small camber angles ϕ , Eq. (8) gives unacceptably low deviation angles, especially in transonic cascades with shock-boundary layer flow interaction. A minimum deviation angle of 2° was arbitrarily assumed and the calculated δ - values were faired gradually to the minimum value toward the tip section. The actual profiles were defined on coaxial cylinders for the most part of the blade. Three profiles were defined on cones in the hub region to ensure a smooth evolution of the profile geometry toward the conical hub section. Fig. 20 shows the relative inlet and exit angles β_1 , β_2 with the tangential direction and the deviation angles δ^2 used to define the cascade geometry. All profiles were set at a nominal incidence $i = +2^{\circ}$ with respect to the suction surface. The selected profile sections are indicated on Fig. 21. Table 5 lists the profile design data defining the cylindrical and conical sections unwrapped on planes tangent to the cylinders and cones. (While all angles are conserved in the development of cylindrical sections, the profile camber angle is reduced in the developed conical sections by the sector angle formed by the radii passing through the leading and trailing edge points.)

The coordinates of the center of gravity of a cylindrical section are determined by the following simple relations:

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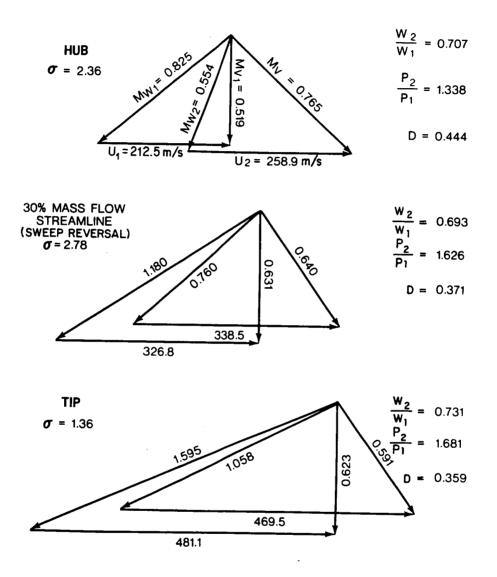


FIG. 19. ROTOR VELOCITY TRIANGLES (28 blades).

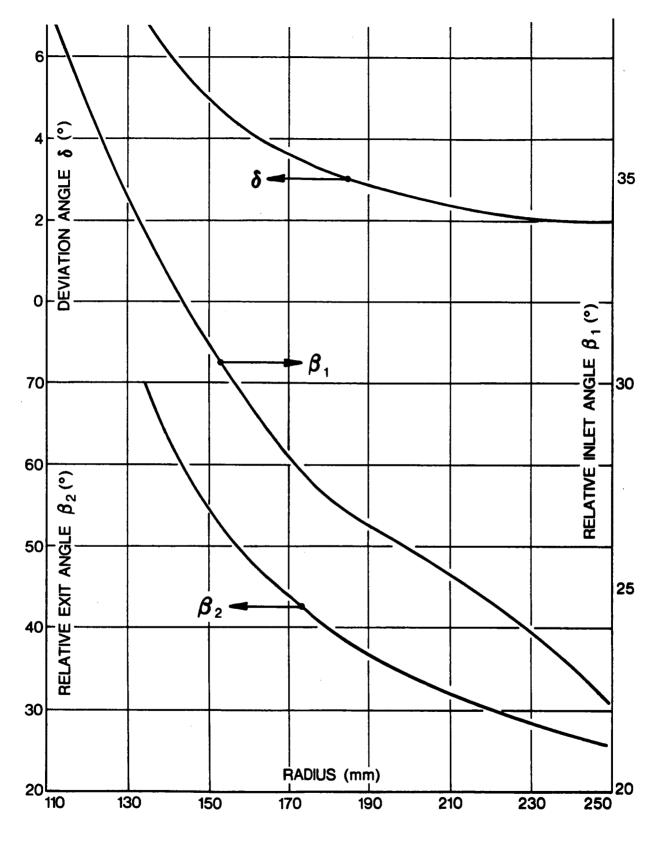


FIG. 20. RELATIVE ROTOR FLOW AND DEVIATION ANGLES.

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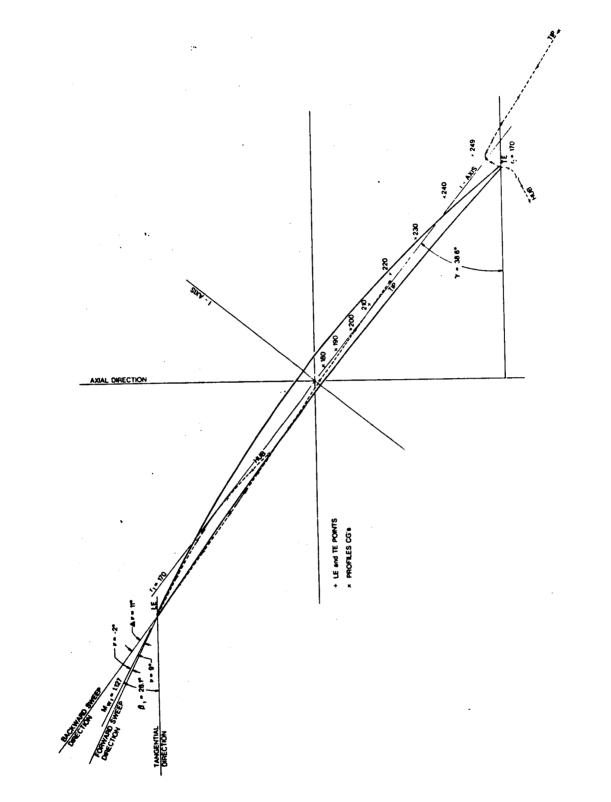




TABLE 5. Rotor Blade Profile Data 28 Blades (Developed Cylindrical and Conical Sections).

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Section Radius (mm)	Mean Camber Angle $\oint (^{\circ})$	Setting Angle	Chord Length c (mm)	Relative Thick- ness ク(%)
110/134	28.85	62.44*	64,55	10.77
122/140	24.70	56.45*	67.91	9.73
136/145	24.95	50.93*	74.73	8.03
150	26.10	46.05	85.20	6.18
160	20.90	41.95	95.20	4.95
170	17.00	38.60	105.90	4.00
180	13.70	36.05	102.30	
190	10,90	33.95	97.70	
200	8.70	32.25	92.90	
210	6.90	30.75	88.20	
220	5.60	29.40	83.80	
230	4.70	28.25	80.20	
240	3.90	27.05	77.20	
249	3.30	25.85	75.00	4.00

*Angle between chord and tangent to the developed section circle at the trailing edge

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$$\theta_{cg} = \theta_{L} + \frac{.5c \cos \gamma + d \sin \gamma}{r}$$
(9)

and

$$Z_{cg} = Z_{L} + .5 c \sin \gamma - d \cos \gamma$$
(10)

where c is the chord length, and d is the distance of the CG to the profile chord in the developed section.

Figure 22 shows the situation for a developed conical section. From the aerodynamic design, the geometric characteristics of the profile, especially the inlet and exit angles β_1 and β_2 between the tangent to the mean camber line at LE g g and TE and the circumferential direction, are known. Also known are the inlet and exit radii r, and r, and the meridional projection c_m of the chord. Hence, from similar triangles in the meridional plane:

$$R_1 = \frac{c_m r_1}{r_2 - r_1}$$
 Further, $R_2 = R_1 + c_m$

and with $m = R_1 \sin \psi$ and $\delta R = R_1 (1 - \cos \psi)$

$$c^{2} = (c_{m} + \delta R)^{2} + m^{2} = c_{m}^{2} + 2R_{1}R_{2} (1 - \cos \psi)$$
(11)

In the developed section, the camber angle is

$$\Phi = \beta_{2g} - \beta_{1g} - \psi$$
(12)

and the setting angle is defined by

$$\sin \gamma = (c_m + \delta R)/c \tag{13}$$

Assuming a circular mean camber line in the developed section, $\beta_{2} = \gamma + \Phi/2$ (14)

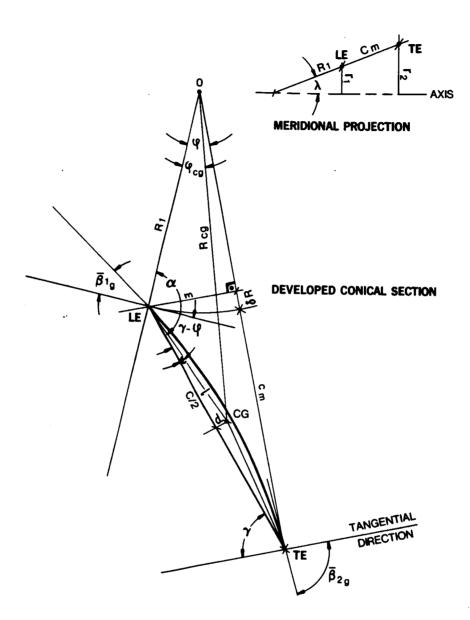


FIG. 22. DEFINITION OF CONICAL BLADE SECTIONS.

Equations (11)-(14) determine the four quantities c, ψ , Φ and γ . They must be solved by successive iterations. Assuming tentatively ψ , equation (11) gives c, equation (12) gives Φ , equation (13) gives γ , while ψ is iterated until equation (14) is satisfied.

After a profile is superimposed upon the circular mean camber line, CG distance d is known and the coordinates of the center of gravity are determined as follows:

 $l^2 = \frac{c^2}{4} + d^2$ (symmetrical profile), $\epsilon = \sin^{-1} (d/l) \alpha = 90 + \gamma - \psi - \epsilon$

Hence, $R_{Cg}^2 = R_2^2 + \ell^2 - 2 R_2 \ell \cos \alpha$ and from triangle O-LE-CG:

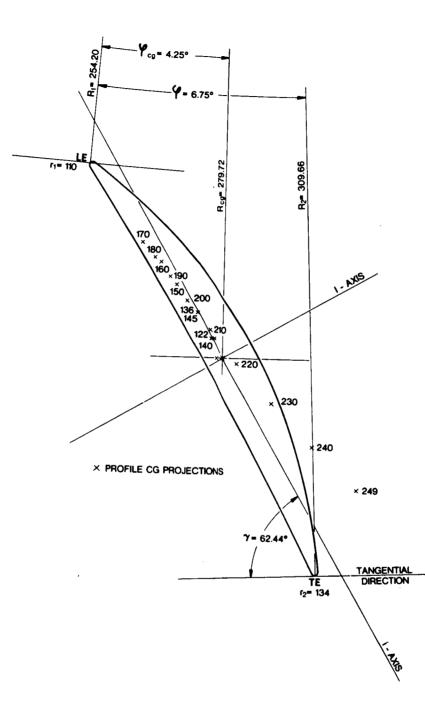
$$\sin \psi_{cg} = l \sin \alpha/R_{cg}$$

Finally, $r_{cg} = \frac{r_1}{R_1} R_{cg}$ and the cylindrical coordinates of the center of gravity are

$$\theta_{cg} = \theta_{L} + \frac{\frac{R_{cg} \psi_{cg}}{r_{cg}}}{r_{cg}}$$
(15)

 $Z_{cg} = Z_{L} + (R_{cg} - R_{1}) \cos \lambda$ (16)

All CG stacking investigations, including preliminary bending stress evaluations, were carried out manually. However, as will be discussed later, verification of stress levels was carried out using computer programs at BBN and AVCO Lycoming. Figure 23 shows the final stacking of the profile CG's radially projected on the conical hub section, which was investigated by NASTRAN analysis. The corresponding distribution of the lateral sweep angle v is shown on Table 4. The NASTRAN results indicated that the stress distribution at the hub section could be improved by a slight tangential shift of the first two conical sections in the rotation sense. $\Delta \theta_{\rm L}$ - shifts of -.008 for the hub and -.004 for the next section were effected without readjusting the z - coordinates of the leading edge points. Those shifts are indicated on Fig. 23. Provision has been made in the i straddling to generate a moment that continuously compensates the moment of the aerodynamic forces, (which are reflected in results hereafter).



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FIG. 23. CONICAL HUB SECTION DEVELOPED ONTO PLANE TANGENT TO CONE, WITH SUPERIMPOSED RADIAL PROJECTION OF PROFILE CG'S.

The optimum radial distribution of the lateral sweep angle v is different for forward and backward leading edge sweep directions. Consequently, a discontinuity of lateral sweep may occur at the sections above and below the point of sweep reversal. This, in turn, results in a high rate of curvature of the blade surface. Since the blade is defined by discrete sections, this appears only as a more or less pronounced concentration of the spanwise curvature of the blade surface in the sweep reversal region. Nevertheless, this local curvature increase generated prohibitive stresses near the trailing edge in several preliminarily generated configurations.

This problem was compounded by the additional bending moment around the I-axis of the section of sweep reversal, due to the rearward location of the CG of the upper blade portion with backward leading edge sweep. The difficulty increases since the sweep reversal was selected initially so as to minimize that moment and it was gradually moved inward from $r_{sr} = 188$ to 170 mm, still leaving the blade CG in a forward position with respect to the I-axis of the hub section. The stress concentration problem at the sweep reversal section was solved by means of an elaborate compromise of the profile stacking through that section, involving especially the selection of the critical lateral sweep angle discontinuity. For the final configuration, with $r_{sr} = 170 \text{ mm}$, this was achieved at a late design state only, the last optimization step, which would have required the sweep reversal point to be set at 160 mm radius, or the profile chord lengths to be increased in the upper blade section. With the present stacking, the highest stress is 645 N/mm^2 (93.5 ksi), which is adequate for concept demonstration purposes. Figure 21 shows the developed sweep reversal section, together with the radial projections of the profile CG's of the upper blade portion and the leading and trailing edge lines. The upper profile CG's have been stacked to compensate for the aerodynamic moment and to minimize the additional TE tensile stress resulting from the rearward CG position of the upper blade portion.

Whereas the radial projection of the leading edge points indicates a smooth subsonic leading edge line, the trailing edge line does not appear to be as smooth as desirable. For manufacturing the blade was defined by flat sections generated from the blade configuration defined in the cylindrical coordinates used for the stacking investigations. Any minor irregularities of the trailing edge were smoothed out by a slight increasing of the chord lengths of a few local sections. All profile data are listed in Table 5.

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5.5 A Review of the Rotor Blade Design Iterations for Stress Optimization

The main objective of the preliminary design effort (see Fig. 24) was to define a stacking configuration that maintains the subsonic leading edge concept, i.e., satisfies the acoustic rotor design requirements with as low a blade stress level as possible. A target design goal of 725 N/mm² (105 ksi) maximum steady state stress was sought for the design speed of 18,450 rpm. For the selected titanium blade, such a stress level is considered adequate for the demonstration purposes of this program.

As a first step in each iteration, both manual and computerized beam-type stress computations were carried out to develop a feel for the iterative stacking procedure and to ensure numerical agreement. The standard AVCO Lycoming blade stress computer program which was used treats the blade as a twisted, rotating cantilevered beam with variable section properties, and takes into account the shroud and aerodynamic forces and the centrifugal restoring moments. All trial blade stacking iterations were analyzed with this program.

Simultaneously, a quick, inexpensive and efficient finite element analysis was used at BBN to verify the results of blade The program, based on SAP, was operated in coniterations. junction with a blade geometry generator which was based on the family of blade profile shapes, described previously by Eqs. 5-7, which reduce to a minimum the number of parameters required to specify a blade shape; namely, the leading and trailing edge coordinates, the section setting angle and camber, and the profile shape parameters. The program was therefore very well suited for iterative design studies. The purpose of the simultaneous effort was to provide further verification of the beam and manual analysis and to help identify stress concentration, which are neglected in the beam-type stress analysis program and in the manual calculations. These efforts were deemed necessary because the blade configuration differs radically from more conventional designs, and it was uncertain whether conventional design methods would be sufficiently accurate.

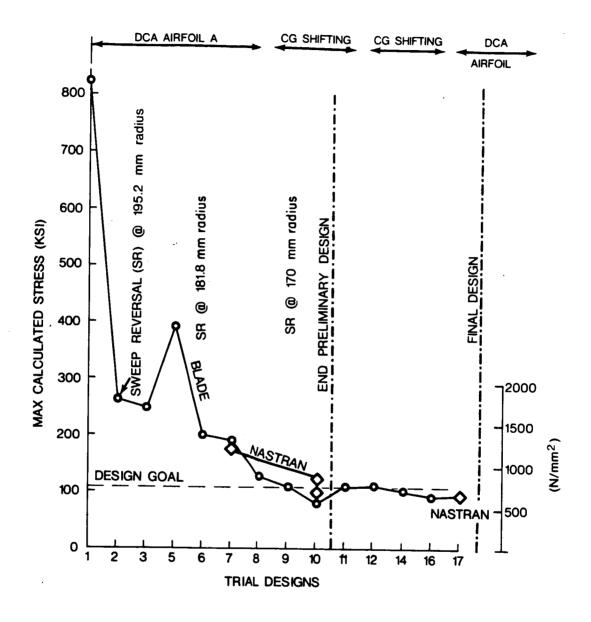


FIG. 24. OPTIMIZATION PROGRESS.

A NASTRAN stress analysis program was used by AVCO Lycoming on design iterations which were considered particularly important, and for the final stress computations verification.

The evolution of the maximum blade stress levels as the blade design evolved through the series of trial designs is shown in Fig. 24. The results of the first design substantiated the impractical stress level of a blade with simple forward leading edge sweep. The initial sweep reversal radius (SR) was selected at 195.2 mm. The stacking for trial design 2 was such that the center of gravity of each of the 13 cylindrical blade sections used to define the blade projected radially down onto the axis of minimum inertia (i-axis) of the airfoil section immediately below. For Iteration 3, all section CG's were projected onto the i-axis of the hub section. For Iteration 5, all section CG's above the sweep reversal section were projected onto the SR section i-axis, while the stacking of Iteration 3 was kept for the lower blade sections. As can be seen, the resulting misalignment of the upper blade portion with respect to the hub section produced higher hub stresses. However, this design also showed the lowest stress level for the upper blade portion.

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For Iteration 6, the sweep reversal radius was lowered and the misalignment was corrected by introducing a discontinuity of the lateral sweep angle, (i.e., the angle between the sweep direction and the radial plane containing the relative inlet velocity), at the point of sweep reversal. By varying this parameter, a number of stacking combinations involving individual compromises within the upper and lower blade sections, were investigated. Iteration 7 shows the best result obtained with this stacking concept.

With the stress level still substantially beyond the preliminary design goal of 105 ksi, a detailed investigation of the stress pattern in design 7 was performed using the NASTRAN stress program. The excellent correlation which was obtained substantiated the beam-theory analysis method as a useful approach to analyze blade stacking changes.

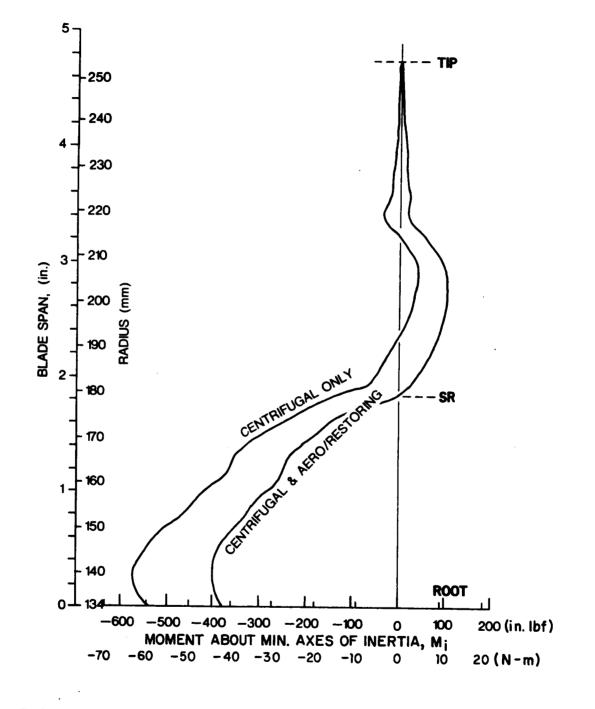
Subsequent iterations were conducted with the optimum stacking concept described in Sec. 5.4. This stacking satisfies the condition that, at every section along the span, the CG of the entire blade portion above the section projects radially onto the i-axis of the section. As shown by Iteration 8, this reduced the maximum stress level very nearly to the preliminary design target value.

The new stacking concept confirmed the necessity of a lateral sweep angle discontinuity at the point of sweep reversal to achieve proper stacking of the profile CG's across that section. This discontinuity, however, resulted in a rapid change of the spanwise curvature of the blade surface in the trailing edge region, which in turn results in a local stress concentration that was not shown by the simplified analysis. Iterations 1-8 were conducted with double circular arc profiles (DCA). Iterations 9 and 10 used new profiles featuring rearward CG shifts from the hub to the section of sweep reversal, and forward CG shifts from that section to the tip (see previous section). In this way, the CG excursions from a radial line were minimized within the leading and trailing edge envelope and the stresses were reduced to the target level.

Figures 25 and 26 show the moments about the axes of minimum inertia and maximum stress distributions for Design 10 as calculated by the standard blade stress program. The influence of aerodynamic loads and centrifugal restoring moments are also shown. (Design 10 was chosen for further study since this is the design which first indicated stresses below the design goal.).

A detailed investigation of Design 10 was also performed with the NASTRAN program. The results showed local high stresses of 96 ksi at the trailing edge of the sweep reversal section and 110 ksi at the leading edge of the hub section. By slightly increasing the chord length of the sweep reversal section, and slight re-alignment of the conical hub, these stresses were brought down to 84 and 96 ksi, respectively. The NASTRAN finite element representation of this configuration, called Design 10A, is shown in Fig. 27. The stress distributions of the suction and pressure surfaces are shown in Fig. 28.

During the entire iteration process, it was apparent that the radial location of the point of sweep reversal would have to be moved substantially inward from its initially assumed location in order to avoid a large moment about the I-axis of the hub section. Moving the point of sweep reversal inboard, however, increases the bending moment about the I-axis of the sweep reversal section, thereby increasing the tensile stress at the trailing edge of that section. To minimize the local trailing edge stress concentration the radial location of the sweep reversal section was moved inboard cautiously. Even so, the blade CG remained ahead of the hub section I-axis, and resulted in an additional bending stress (on the order of 120 N/mm²) at the hub section leading edge.



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FIG. 25. SECTION MOMENT DISTRIBUTION [Preliminary Design 10].

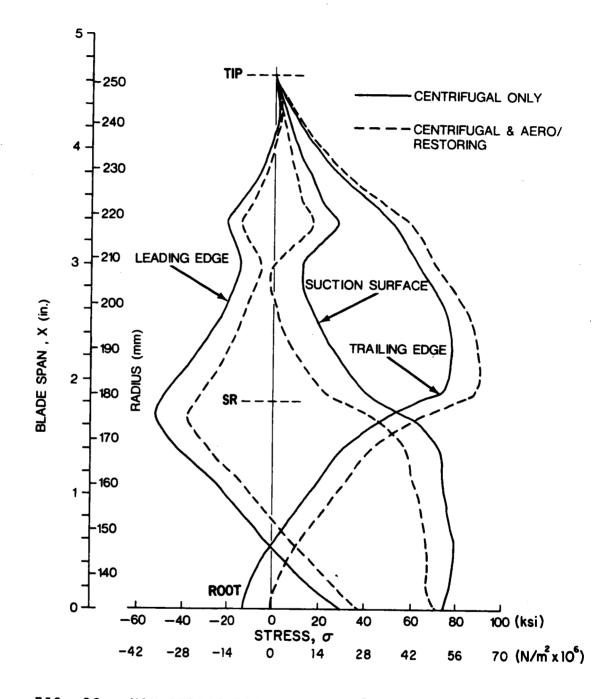
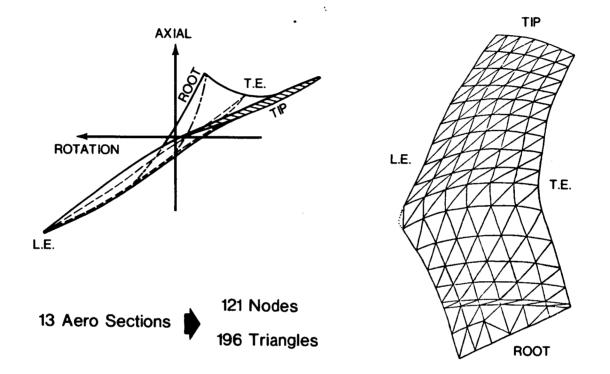


FIG. 26. MAX STRESS DISTRIBUTION [Preliminary Design 10].



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FIG. 27. NASTRAN ANALYSIS [Preliminary Design 10A].

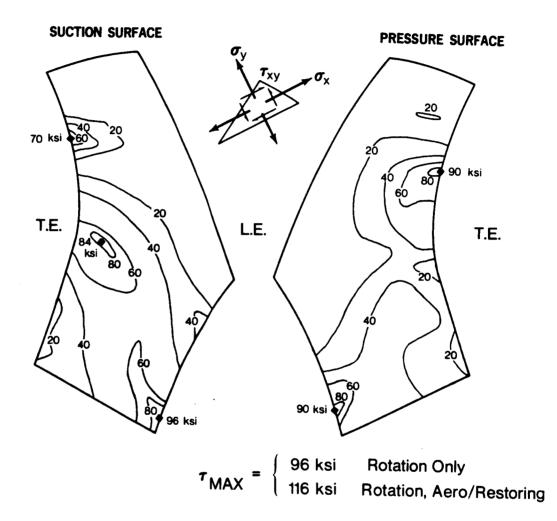


FIG. 28. EXAMPLE OF INTERIM RESULTS OF NASTRAN STRESS ANALYSIS MAX SHEAR CRITERION [Preliminary Design 10A].

Prior to the selection and analysis of the final blade design, several intermediate designs were investigated based on local shifts of the CG location within the indificual airfoil profiles. (Noted as Designs 11 through 16 in Fig. 24.) The polynomial blade sections had been evolving toward n=1 or a DCA profile. For manufacturing reasons, however, double circular arc profiles were specified for the final design. This raised the stresses to virtually the level of iteration 9, and additional stacking iterations were required to achieve the design objective. In particular, the relative blade thickness was increased from 10.00 to 10.77% at the hub section. This resulted in a 10% decrease in the stress level. Additional reductions were achieved through a judicious balancing of the profile stacking in the lower blade portion and lateral sweep angle discontinuity at the sweep reversal section.

A check was performed to see if the DCA profiles allowed adequate flow area margins. On an average basis, the rotor throat passage area has a large margin to sonic throat area because of the comparatively high mean relative inlet Mach number level $\overline{M}_{W_{i}}$ = 1.33 and the positive inlet incidence of 2° selected for optimum blading efficiency. The throat hub region is most susceptible to local throat choking because of the transonic inlet flow conditions and the higher relative blade thickness. Because of unknown 3-dimensional flow effects, it is difficult to determine local blade stream tube areas and no definite section throat area margins thus were specified for the design. A check, however, was tentatively made for the rotor hub section. On the two-dimensional basis of the developed section of Fig. 23 the ratio of throat to inlet passage width is 1.045. At the throat location, however, the channel height has decreased from 139 to 136.3 mm. Assuming that all individual stream tube heights are reduced in the same proportion, the effective geometric throat/inlet area ratio thus is Amin/Ain = $1.045 \times 136.3/139 = 1.027$. With a relative inlet Mach number of .825, the sonic area ratio A_{in}/A_s is 1.0285, thus A_{min}/A_s = 1.027 × 1.0285 = 1.055, i.e., a 5.5% choke area margin.

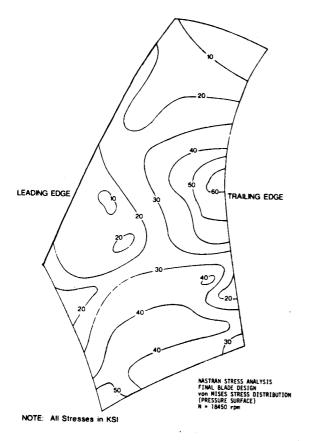
In the hub region, the flow has the tendency to be deflected inwards because of the forward leading edge sweep. On the other hand, the increasing density toward the tip at rotor exit combines with the essentially constant axial velocity of free-vortex flow to shift the streamline pattern outwards at rotor exit. Those compensating effects cannot be quantified at the throat location and the comparatively large 5.5% margin thus was judged adequate to account for the possibility of unfavorable threedimensional effects and for the suction side boundary layer growth upstream of the throat in the absence of a detached leading edge shock. In summary, in spite of the selection of DCA profiles, the individual rotor section throat margins are adequate.

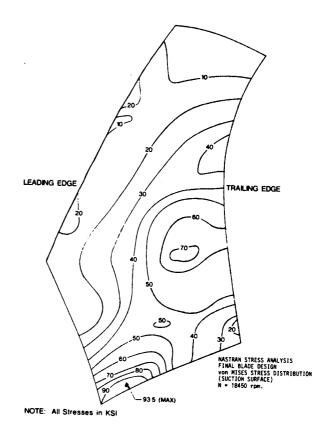
5.6 Final Rotor Blade Stress Analysis

The stress analysis for the final design iteration was performed using NASTRAN. The loads considered in this run were based on the maximum operating speed of 18,450 rpm. In addition to the major contribution of the centrifugal load, aerodynamic gas pressure loads, the centrifugal load and the torsional restraints of the part span shroud were applied to the blade. The resulting von Mises effective stress patterns over the pressure and suction surfaces of the blade are shown in Figs. 29a and 29b. An independent verification of these results was performed using the SAP program at BBN.

The maximum stress level of 645 N/mm^2 (93.5 ksi) is at the root near the leading edge on the suction surface. The high stress region of 90 ksi, however, extends only over a small portion of the suction surface (Fig. 29b) and so should not pose a problem for the planned test program. The permissible number of start/maximum speed/stop cycles is approximately 500, considering a notch condition (SCF = 3.5) at the juncture of the blade airfoil and the base shroud.

The tendency of the blade to untwist at the shroud location is small since there is only 1/2 degree difference in untwist between the shrouded and unshrouded NASTRAN results. The most significant load on the shroud, therefore, is the bending load due to the centilevered mass. The maximum shroud stress of 78.7 ksi is at the blade-shroud juncture, and is conservative in that the large fairing radius at the juncture was not included in the calculation. Because of the constraining effect of the mid-span shroud, the untwist of the blade at the shroud location is negligibly small. The untwist of the tip section calculated from the NASTRAN results is .36°, thus increasing the tip incidence from 2 to 2.4° at the design speed, a value well within the blade incidence design tolerance. However, radial growth of the shroud has not been accounted for and, if such growth occurs, undesirable increased tip incidence angle could result, due to the consequences of shroud sections "unlocking".





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FIG. 29a.

Pressure surface

FIG. 29b.

Suction surface

FIG. 29 NASTRAN STRESS ANALYSIS: FINAL ROTOR BLADE DESIGN

The magnitude of the stresses in the fan blade airfoil are acceptable for an experimental program. The computed stresses in this design exceed AVCO Lycoming practice for titanium blades for longtime service operation, but fall within acceptable limits for the planned experimental program.

5.7 Rotor Blade Vibration and Flutter

The avoidance of large amplitudes of resonant vibration of the rotor blades over the full operating range is necessary to ensure the structural integrity of the fan. The design procedure included an assessment of the natural frequencies of the rotor blade so that the forced vibration response is minimized, and the self-excited response is eliminated. The design goal for the minimization of forced vibration is ensuring that the rotor blade cannot resonate with the first three rotational orders of excitation due to possible inlet distortions. Although higher ex-citation orders will exist in the intake, it is considered that these levels will be minimal in the clean inflow expected in the acoustic test facilities and, thus, they will not generate significant resonant stress levels in the blade. The avoidance of self-excited blade vibration flutter is mandatory, since the associated stress levels usually lead to blade failure in a very short time. The two flutter phenomena that were considered in the design are subsonic positive stalled flutter at part-speed operation and supersonic unstalled flutter at design speed. The criteria for avoidance of these flutter conditions are based on extensive experience by the engine manufacturers and are expressed in terms of a reduced velocity parameter: $u/b\omega$, where u =air velocity over the blade (m/sec), b = blade semichord (m), and ω = frequency of vibration in the flutter mode (rads/sec). The empirical design limit values for this parameter under positive stalled flow are 6.7 and 2.4 for the first bending and first torsion modes, respectively. The supersonic unstalled flutter design limit at first torsion frequency was:

 $\frac{u}{b\omega}$ $\left(\frac{M^2-l}{M}\right)$ < 1.05, where M = Mach number.

The coefficients are calculated at 3/4 span. (Since supersonic unstalled flutter usually occurs in vibration modes which are predominantly torsional, only this mode is considered.)

A free-standing blade, assumed fixed at the base, was used in the calculation of the resonant frequencies. The natural frequencies for the unshrouded blade are shown in the excitation diagram of Fig. 30. This design is clearly unsatisfactory since the natural frequency of the first bending mode has a second order resonance in the operating speed range. The stall flutter coefficients are 3.44 and 1.53 for bending and torsion, respectively. These values are within the safe limits which were established as design criteria. The supersonic unstalled flutter parameter is 1.16 and exceeds the safe upper limit.

A partspan shroud is required to raise both the first bending and torsion natural frequencies and avoid forced and self-excited vibrations (flutter). As a physical model, the shroud was assumed to restrict the blade motion to a uniform translation at three representative points. 1

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The design analysis was checked by mounting two spare blades in a fixture which clamped at the root and partspan shroud locations. An acoustically coupled exciter was used to vibrate the blade so that the frequencies and mode shapes could be obtained. The comparison between the measured and theoretical static frequencies shown in Fig. 31, is considered good, especially in view of the unusual blade shape. The "measured" frequency line in Fig. 31 is actually the theoretical centrifugal stiffening line originating at the measured static frequencies of the first three modes.

Figure 31 shows the excitation diagram and calculated and measured frequencies for the final airfoil with the partspan shroud located at 64% of the span (201 mm radius). The first bending natural frequency has been raised so that it clears the first three excitation orders in the operating speed range. The fourth excitation order of the first mode, (e.g., four equally spaced front struts) however should be avoided. Based on the measured frequencies, the stalled flutter coefficients are 1.5 and 1.0 for bending and torsion, respectively. The supersonic unstalled flutter coefficient is .75. These values meet the design criteria. The excitation diagram shows that the torsion and bending modes are not coincident in the operating speed range. This ensures that the modes are decoupled.

Strain gauges will be used during the test program to ensure that safe steady and vibrating stress levels are not exceeded. In order to locate the strain gauges appropriately, a vibratory stress survey was conducted using strain gauges during the static vibration tests: Fig. 32 shows the results of this test, normalized for each mode. The vibratory stress distributions, shown as

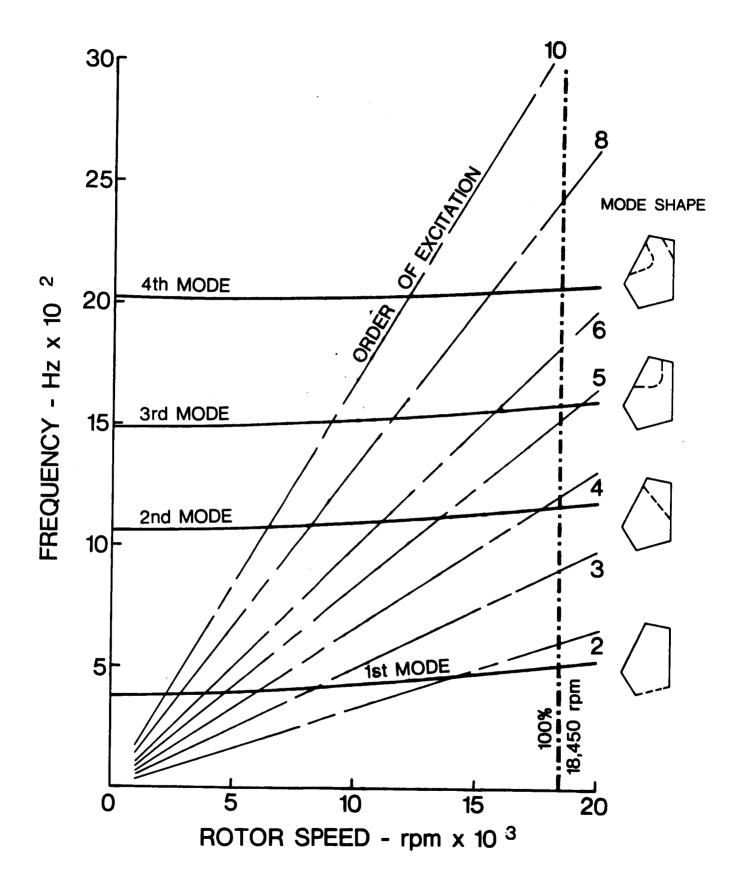


FIG. 30 RESONANCE DIAGRAM OF FINAL BLADE BEFORE SHROUD WAS ADDED.

PARTSPAN SHROUD LOCATED AT 64% SPAN (r = 201 mm)

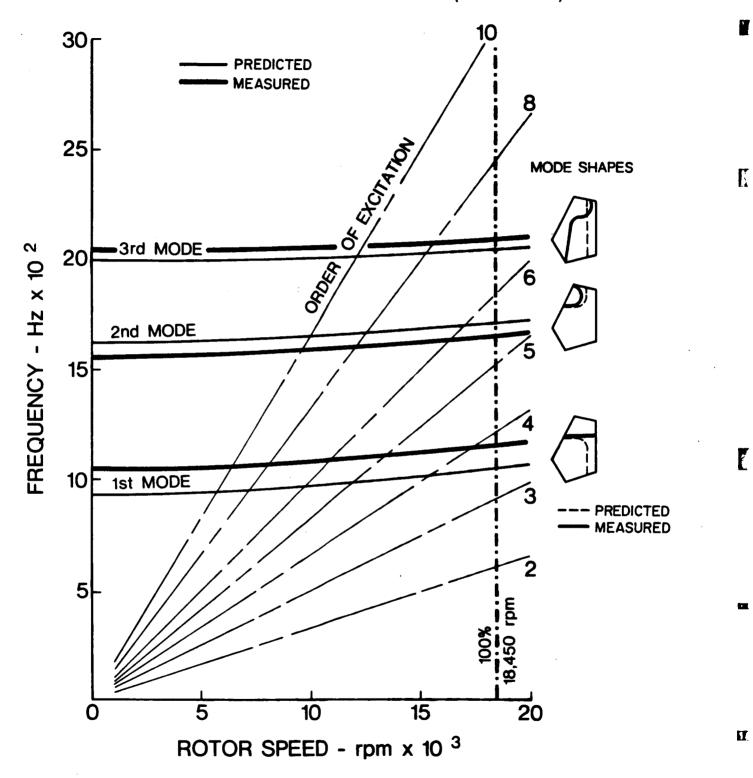
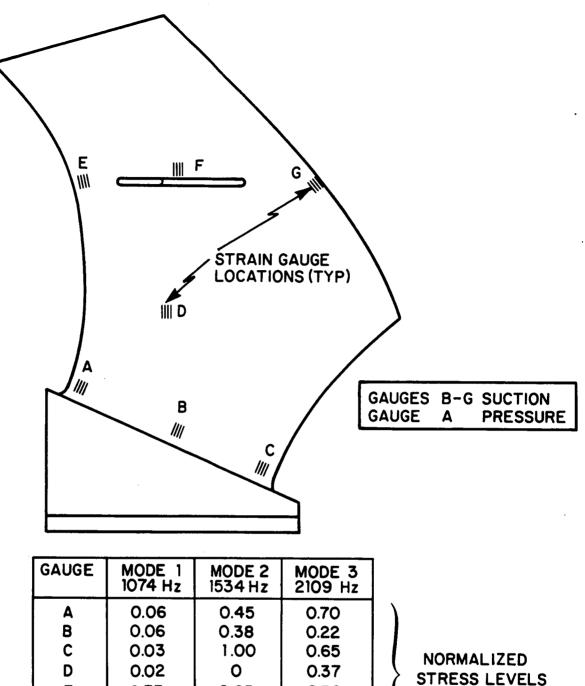


FIG. 31 RESONANCE DIAGRAM OF FINAL (SHROUDED) BLADE. 68

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STRESS LEVELS

FIG. 32. MEASURED AND NORMALIZED STRESS DISTRIBUTIONS DURING STATIC VIBRATION TESTS ON BLADE S/N 17.

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0.18

0.30

1.00

0.43

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0.33

1.00

0.13

the combined steady and alternating stresses in the blade, are plotted for each mode in conjunction with calculated steady stresses at each gauge location used. Figure 33 shows the Goodman diagram for the blade material and the vibrating stresses measured in each mode proportioned for the most critical location. From this diagram it is seen that location 'F' is the most critical location in terms of combined stress in the first mode of vibration. Location 'C' is seen to be the most critical for the second and third modes of vibration. It is therefore recommended that strain gauges at positions 'C' and 'F' are used to monitor the steady and vibrating stresses during the rig running.

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5.8 Attachment and Disk Analysis

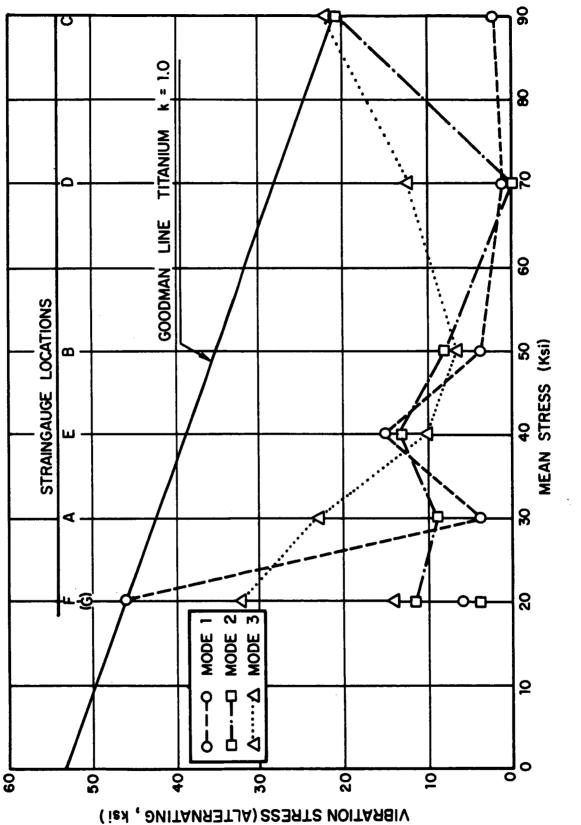
The fan disk stresses were computed by a Lycoming finite element program which evaluates the loading variation throughout the disk accounting for the effects of rotation, temperature gradients and elastic-plastic conditions.

Low cycle fatigue (LCF) life was evaluated for the significant regions, i.e., the disk serrations, the bolt holes and the disk bore, utilizing statistical minimum fatigue property data for Timkin 17-22AS material. The stress/strain ranges utilized in the life evaluation are the stabilized values corresponding to start/stop excursions to 18,450 rpm design speed.

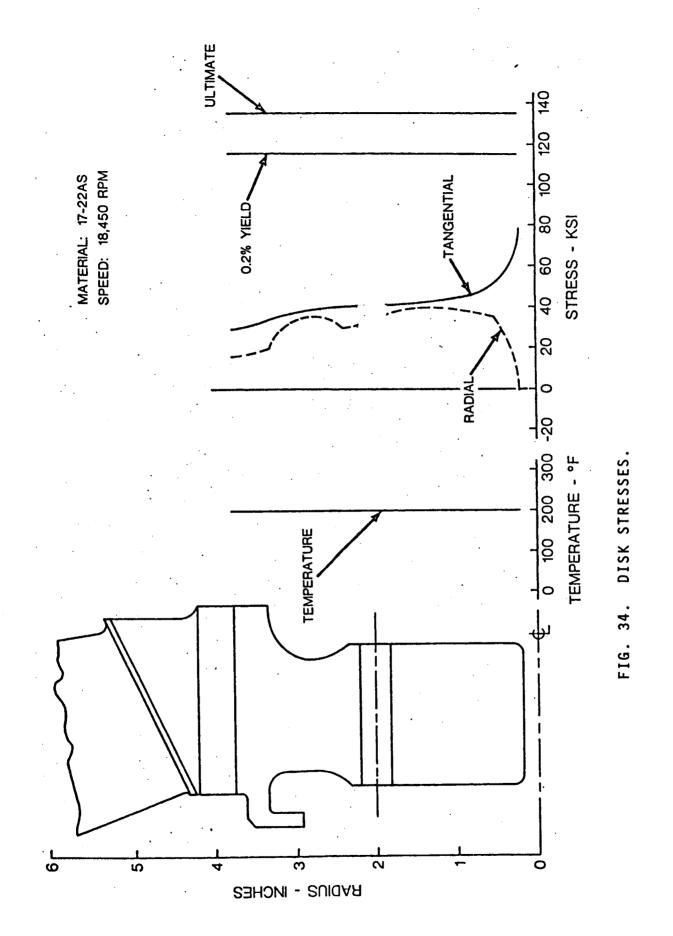
Stress concentration factors (SCF) were evaluated for those areas of the disk containing a high stress gradient, i.e., the serrations and bolt holes. This was accomplished by ratioing the peak stresses determined by finite element analyses with the nominal stresses in each of the two regions.

Nominal radial and tangential stress distributions for the fan disk are shown in Fig. 34, while the nominal stresses in the servation are shown in Fig. 35. The finite element models for the bolt holes and servations are handled separately.

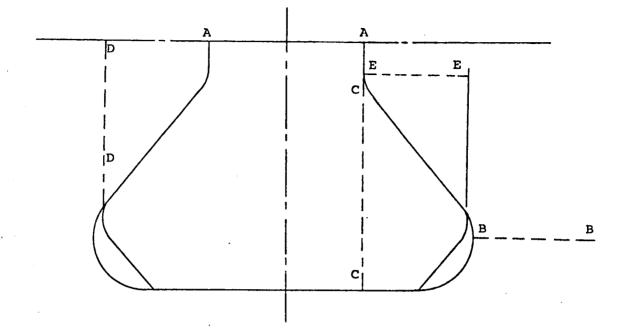
Stress distributions about the disk bolt-holes are given in Fig. 36 from which an SCF of 2.06 was calculated, so that the resulting LCF life is in excess of 100,000 "start/stop" cycles based on the material S-N data of Fig. 37. It has been concluded that the disk bore also has a calculated life of at least 100,000 cycles.



DETERMINATION OF CRITICAL VIBRATORY STRESS LOCATIONS. (Shrouded Blade). FIG. 33.



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	.		Type of	Stress	Yield S Disk (ksi) - 115,000 57,500	Yield Strength	
Ì	Part	Location	Stress*	Level	Disk	Blade	
`				(ksi)	(ksi)	(ksi)	
	Blade	A-A	Tensile	49.20	-	100,000	
		C-C	Shear	21.00	1	50,000	
4 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1	Disk	В-В	Tensile	53.07	115,000	-	
		D-D	Shear	17.36	57,500	-	
		E-E	Bearing	61.03	-	-	

*Includes C.F. and Bending Effect

A State States

Blade Material:	'Titanium 6AL-4V	Temperature:	200 ⁰ F
Disk Material:	17-22AS	Speed: 18,45	D rpm

FIG. 35. DISK/BLADE ATTACHMENT STRESSES.

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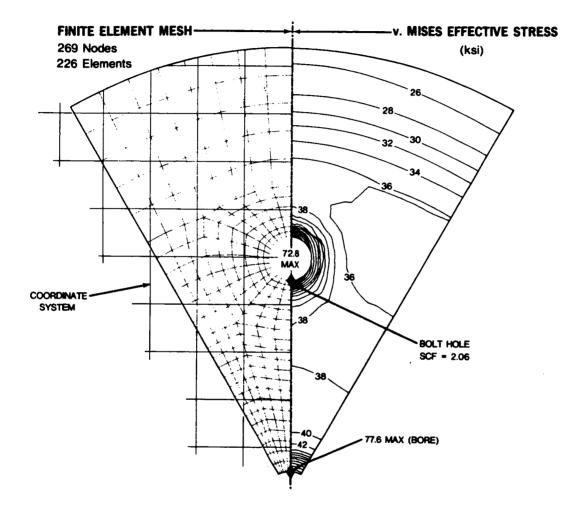
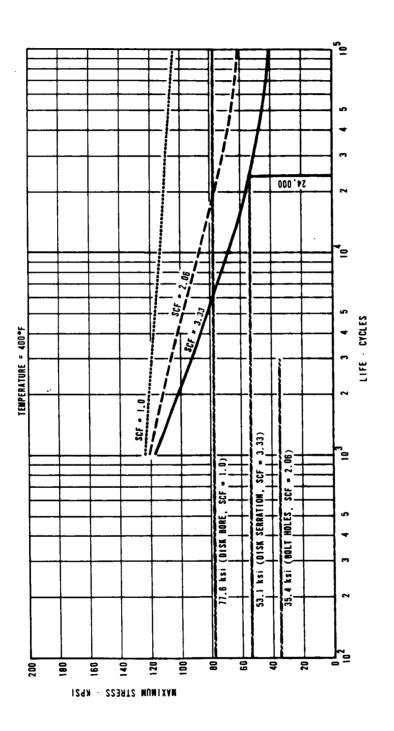


FIG. 36. DISK FINITE ELEMENT STRESS ANALYSIS.



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Blade root attachment stresses and corresponding material properties are also summarized in Fig. 35. The bending effects in both the root and tenon have been included. The axial width of the base-shrouded dovetail root was determined by a permissible bearing stress of 420 N/mm^2 (61.0 ksi). This is less than the compression yield limit, yet somewhat beyond the level at which fretting can occur under prolonged operation, but which should be satisfactory for a limited experimental program.

From a disk servation finite element analysis, the SCF was calculated to be 3.33 which is consistent with values measured from photo-elastic analyses of similar blade root configurations. The corresponding LCF life is 24,000 "start/stop" cycles based on the appropriate curve of Fig. 37. These fatigue lives are ample for the anticipated program of testing.

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SECTION 6

DETAILED STATOR DESIGN

This section describes the detailed design of the fan stator which embodies the stator noise reduction concept described earlier. The stator uses vanes with varying sweepback angle to meet the criterion of a constant subsonic rotor wake trace speed along the stator vane span. The use of circumferential vane skew (lean) was avoided primarily to simplify the manufacturing problem. The stator vane number was chosen to cut off the radiation from residual sources due to end effects in the hub region of the vanes at blade passage frequency. The corresponding residual sources at the tip cannot be cut off because the spinning speed of wake disturbance pattern is supersonic at the tip.

To determine the proper vane sweep angle distribution, the rotor wakes were assumed to be convected with the mean flow. The spatial location of the wake centerline surfaces could then be computed from the mean flow properties by integration downstream from initial points on the rotor trailing edge. Since the rotor wake pattern spins fixed with respect to the rotor, it is possible to find leading edge lines whose shape is such that their point of intersection with the rotor wake centerlines travels at constant speed. Moving medium effects were taken into account in the actual calculation of a vane leading edge shape (see Appendix C for details). The trace speed was made constant and subsonic relative to the local flow velocity vector at all points on the vane span. The stator vane sweep distribution was designed to have an effective spanwise trace speed corresponding to a Mach number of 0.8 for the traveling load distribution.

The fundamental acoustical analysis which underlies the stator design concept is presented in Appendix C. In the remainder of this section, the methods for determination of the vane leading edge shape, and vane number are described, and the aerodynamic design considerations for the stator are reviewed.

6.1 Acoustic Aspects of Stator Design

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The major noise producing mechanism of the stator is the interaction between the stator blades and the wakes shed by the rotor. This interaction causes fluctuating lift at the stator blades; the fluctuating lift in turn can be a potential source of noise. The fluctuating lift is restricted essentially near the stator blade leading edges (SBLE); this fact is made abundantly clear from the analytical work of Filotas (Ref. 10). It is also well known (e.g., Lighthill, Ref. 15) that any fluctuating lift, whether at the leading or trailing edge, whether acoustically compact or not, whether in a stationary or moving acoustic medium, acts as a dipole source of sound.

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However, irrespective of the *nature* of the sources (i.e., whether monopole, dipole or quadrupole, etc.), there are certain aspects of acoustics of stationary and uniformly moving media which need to be considered before approaching the specific task of stator design and related acoustic problems. Discussion of these fundamental aspects is provided in Appendices C.1 and C.2, and their application to the stator design of this fan is described below.

6.1.1 Criteria for non-radiation

Acoustic wavelengths at rotor blade passage frequency are small compared to the stator blade span. In this case, the criterion for non-radiation due to unsteady forces is that the trace phase velocity of the force disturbance be subsonic relative to the local gas flow. Skewing, or sweeping of the stator blade, increasing the separation between rotor and stator, and shaping the rotor blade are techniques which can be used to reduce the phase trace speed.

Proper modification of leading edge profiles can reduce the phase trace speed along the leading edge and also the relative angle between that velocity and the local flow. Both effects are important as it is the trace velocity relative to the local gas-properties flow which must be kept subsonic.

Each individual wake shed by a rotor blade suffers a lag in the circumferential direction. The net effect of this lag on the nature of impingement of the wake on an unswept SBLE is that the wake hits the SBLE at the hub first and the impingement process propagates radially outwards towards the SBLE tip with a spanwise varying phase or trace velocity $c_0(r)$. Sweeping back the SBLE enhances this phase lag effect, in the sense that the spanwise trace velocity of wake impingement is reduced. A criterion along the lines of Eqs. (C.62) and (C.63) is used to guarantee that the wake trace Mach number m_0 is less than m_u everywhere along the swept back SBLE. The nature of the wake phase lag and the calculation of the SBLE sweep angle is described more fully below. The successful analysis of a rotor wake tracing along the leading edge of a stator requires understanding of a set of transformations between stator-fixed co-ordinates and moving medium coordinates. The derivation of the trace velocity in stator-fixed coordinates, and subsequent Gallilean transformations to gas-fixed coordinates is given in Appendix E.

6.1.2 Estimate of rotor viscous wake

Estimates of the magnitude of the rotor viscous wake at the leading edge of the stator have been made. The method of estimation involved modeling the rotor blade wakes as the wakes behind isolated airfoils. The method is somewhat crude, as it ignores the interference between wakes and the axial pressure gradient. The variation of angle of attack at the stator which results from the estimated velocity fluctuations is as much as 10 degrees from the mean. Experience with axial flow turbomachinery wakes indicates that the estimated rotor wake amplitudes at the stator leading edge are likely to predict higher resultant angle of attack fluctuations than will exist in the actual rotor wake. This is due to the higher rate of decay of rotor blade viscous wakes in turbomachines when compared to isolated viscous wakes in free flow (see, for example, Lakshminarayana and Raj; Ref. 16).

6.1.3 Computation of rotor wake distortion

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Contours of constant phase for rotor wakes at different axial locations were computed by use of a stepwise integration of the phase lag of the wake relative to a point in the rotor, as a function of radius. Cylindrical helical flow was assumed (radial flow velocity was assumed to be zero). Axial and tangential velocities used in this calculation were provided by AVCO's aerodynamic design program (Appendix A). Contours of constant phase calculated at several stations downstream of the rotor are shown in Fig. 38 (see Fig. 14 and Appendix A for Station Locations). Contours of constant phase versus axial location on cylindrical surfaces, shown in Fig. 39 and contours of constant phase in the axial/radial plane, shown in Fig. 40, were derived by cross-plotting from Fig. 38.

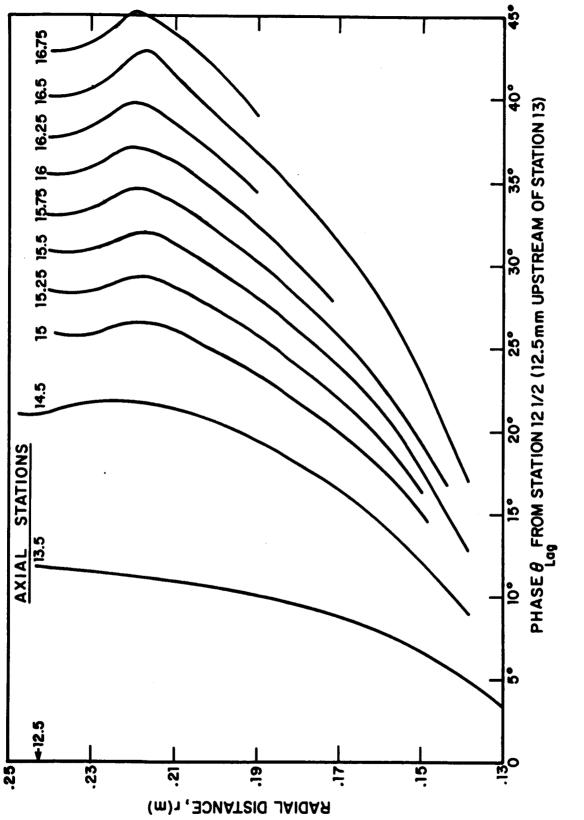


FIG. 38. PHASE vs RADIUS AT DIFFERENT AXIAL POSITIONS.

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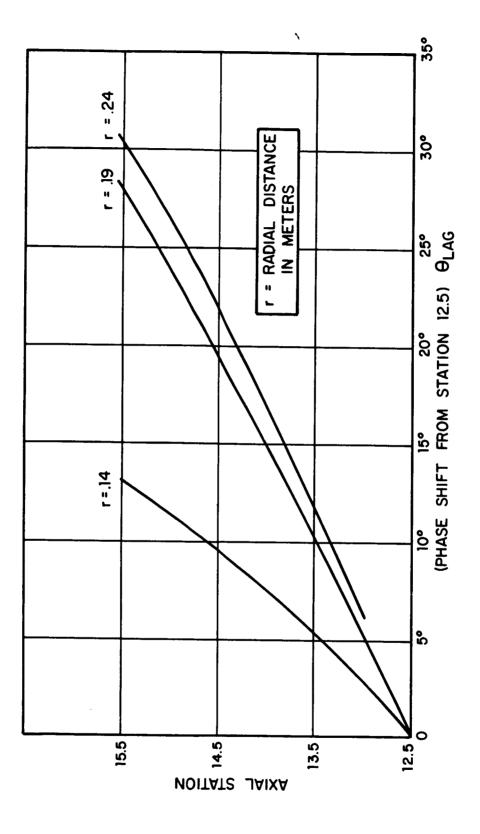
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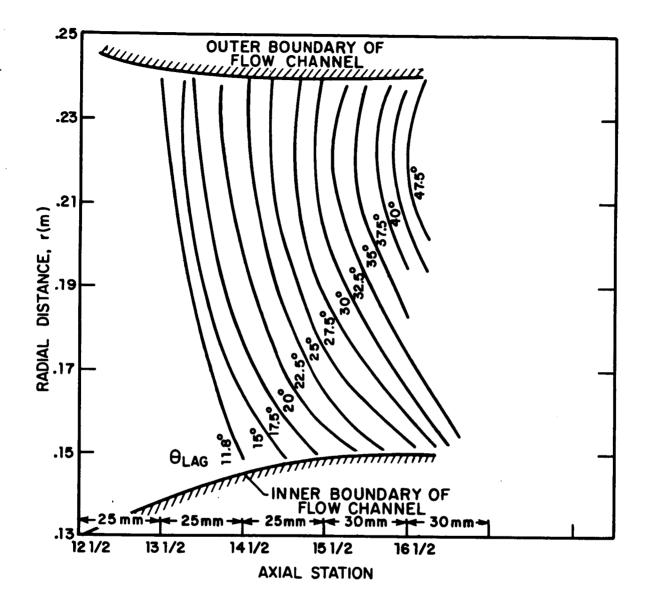


FIG 40. CONTOURS OF CONSTANT PHASE IN Y-Z PLANE.

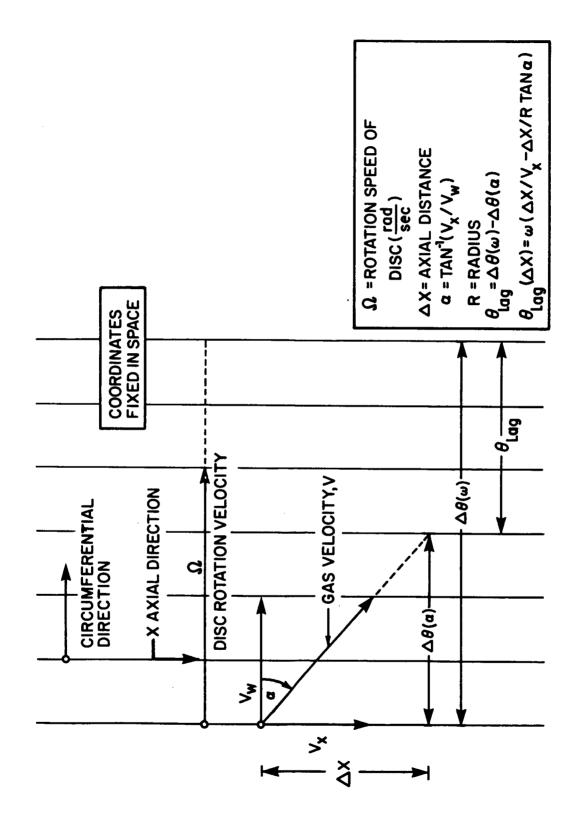
Figure 41 illustrates, in stationary coordinates, the flow and blade motion geometry and the equations used in the calculation of the constant phase contours. The two terms $\Theta(\alpha)$ and $\Theta(\omega)$ in the calculations represent the angular translation of a fluid particle and the angular rotation of the rotor, respectively, in the time required for the fluid particle to flow from axial Station 1 to Station 2. The trace velocity, relative to local flow, for a number of constant-sweep-angle stators is shown in Fig. 42 The very high trace velocities near the tips result from reduced wake "windup" in that region, thus illustrating the limited effectiveness of constant angle swept stators.

6.1.4 Mach .78 leading edge stator

A blade leading edge sweep profile for trace speeds less than Mach 0.8 was developed for the final rotor and flow path design using an iterative method to achieve a nearly uniform trace velocity. The blade has a minimum sweep angle of 25 degrees at approximately 1/3 of the span from the root. Sweep at the root and tip are 30 and 40 degrees, respectively. Figure 43 shows the sweep profile as well as the trace and acoustic speeds as a function of radius.

The calculations assume a rotor blade reference axis at the axial location 12-1/2 mm. forward of the root at Station 13. It also assumes a stator leading edge which is radial, when projected in the r, θ plane, and has its root 12.5 mm. downstream of Station 15.

The inflow-induced radiation from an array of such variablyswept stator vanes is now restricted to the tip regions when discontinuities occur. The limitation of such radiation depends upon proper choice of the number of swept leading edge stators, which in turn depends upon the rotor blade number, rotation speed, and moving medium acoustical considerations. The computation of vane numbers is discussed below.





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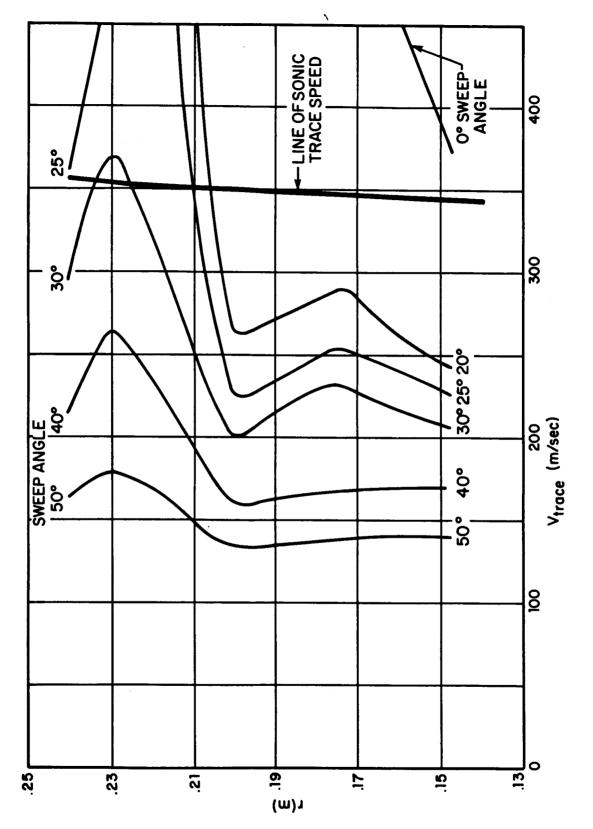
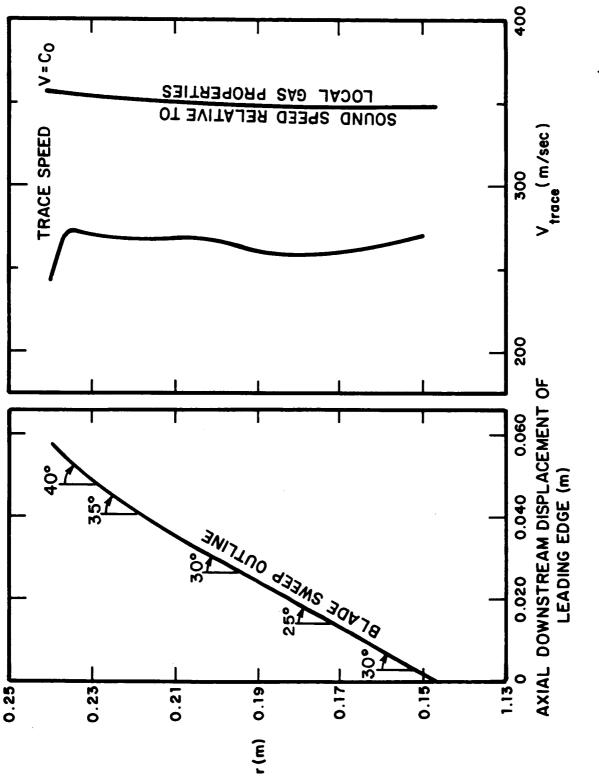


FIG. 42. TRACE VELOCITY FOR DIFFERENT SWEEP ANGLES.



MACH.78 LEADING EDGE PROFILE (FINAL VANE DESIGN). FIG. 43.

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6.2 Analysis For Determination Of Number Of Stator Blades

This section determines the appropriate minimum number V of stator blades so that the acoustic noise radiated from the stator at the rotor blade passage frequency f_r is minimized.

Since the swept back SBLE derived in the above section is of finite extent, the end effects at the SBLE hub and tip from the wake/SBLE interactions remain as potential sources of noise. The aim here is to seek a partial *circumferential* cancellation of these end sources. Thus, two discrete circumferential arrays exist, one at the SBLE hub and the other at the SBLE tips. Since the circumferential phase velocity c is higher at the tip than at the hub, one concentrates on the discrete circumferential array made up of uncancelled sources at the SBLE tips. Also, as discussed in Appendix C, the discussion of a discrete array must be limited to only one frequency ω_0 . Choosing ω to correspond to the fundamental rotor harmonic, (i.e., to ^O the rotor blade passage frequency f_r), one obtains

$$\omega_{0} = \Omega B$$
, (17)

where Ω is the shaft rotation in radian/sec ($\Omega \gtrsim 1940$ rad/sec) and B is the number of rotor blades (B = 28). The blade passage frequency f_r is given by

$$f_r = \frac{\omega_o}{2\pi} \approx 8600 \text{ Hz}$$
(18)

For the circumferential phase velocity c

$$c_{o} = \Omega r_{t} , \qquad (19)$$

where r_t is the radius at the stator tip ($r_t \gtrsim 0.24m \gtrsim 0.79$ ft). The corresponding Mach number m is therefore given by

$$m_{o} = \frac{c_{o}}{c} \approx 1.27$$
 (20)

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With reference to results of Appendix C.2, (Eq. C.60), m_1 is the Mach number of the gas flow parallel to the array, and m_r is the gas Mach number normal to the array. Since the array^r under consideration is oriented circumferentially, the gas Mach number m_r in the circumferential direction plays the role of m_1 and the gas Mach number m_r in the axial direction plays the role of m_r ; the radial gas ^aMach number, normal to the duct walls, is to a good approximation zero. Thus, we have

$$m_1 \equiv m_2 \approx 0.353 \tag{21}$$

$$m_{r} \equiv m_{a} \approx 0.582 \tag{22}$$

Note that m_c is directed the same way as the shaft rotation Ω or the phase Mach number m. Hence, first one would like to find from Eqs. C.62 and C.63 whether $m_0 < m_u$ - one of the two necessary conditions for no radiation to occur. Substituting the quoted values in Eq. C.62, one finds that

$$m_0 < m_u$$
, for $\frac{\pi}{2} < \alpha \leq \frac{\pi}{4}$, (23)

in other words, the condition for no radiation is satisfied for angles α that are sufficiently remote from the axial direction.

In order to satisfy the second condition for (partial) cancellation of radiation from the fundamental rotor harmonic, it is required that

$$\frac{2\pi}{d_{t}} > 2 \frac{2\pi}{\lambda_{r}} \frac{(1 - m_{a}^{2} \cos^{2} \alpha)^{1/2}}{(1 - m_{c}^{2} - m_{a}^{2} \cos^{2} \alpha)} , \qquad (24)$$

where d_t is the circumferential spacing between two adjacent SBLE tips. The spacing, d_t , is related to the number of stator blades by the relation

$$d_{t} = \frac{2\pi r_{t}}{V}$$
(25)

The right hand side of Eq.(24) is the radiation span $\begin{pmatrix} k_a & -k_a \end{pmatrix}$ obtained from Eq. C.60, where $2\pi/\lambda_p$ has been o+ o- substituted for k_a , λ_p being the acoustic wavelength at frequency f_r . o Taking sound speed c in the gas to be about 365 m/s (1200 ft/sec), the wavelength at blade passage frequency is

$$\lambda_r = \frac{c}{f_r} \approx 0.14 \text{ ft} \approx 0.043 \text{ m}$$
(26)

Thus, substituting Eq. 25 in Eq. 24, the velocity is

$$V > \frac{4\pi r_{t}}{\lambda_{r}} = \frac{(1 - m_{a}^{2} \cos^{2} \alpha)^{1/2}}{(1 - m_{c}^{2} - m_{a}^{2} \cos^{2} \alpha)}$$
(27)

Since the first necessary condition (Eq. 23) is satisfied only for a restricted range of angles α , it would not pay to find the maximum possible value of V for arbitrary α . Instead, Eq. 27 is evaluated for $\alpha = \pi/4$ (as α goes from $\pi/2$ to $\pi/4$ to 0, V evaluated from Eq. 27 increases), and the result is

$$V \ge 92$$
 (28)

One can now examine the application of traditional analyses (e.g., Tyler and Sofrin (Ref. 11)) of noise generated by rotorstator interaction, the anlysis that is used primarily for lowspeed compressors (i.e., analysis is based on stationary medium acoustics) that involve subsonic circumferential phase speeds (i.e., $\Omega r_{+} < c$) and are acoustically compact (i.e., $d_{+} < \lambda_{p}/2$).

An arbitrary component (say, the predominant component of wake velocity deficit pattern that generates fluctuating lift at SBLE) $a(x,r,\theta,t)$ of rotor-generated flow field near the stator may be decomposed into circumferential harmonics as follows

$$a(x,r,\theta,t) = \sum_{n=-\infty}^{+\infty} A_n(x,r) e^{i[nB(\theta-\Omega t)]}, \quad (29)$$

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where x and r are the axial and radial locations (and for our case of interest denote the locations of SBLE tips) and θ is the circumferential angle.

The noise sources (in particular, the fluctuating lift l generated at SBLE tips) at the stator due to the nth rotor harmonic may then be viewed as composed of a sum of stator/rotor harmonics mn. A typical interaction harmonic $L_{mn}(s,r,\theta,t)$ may be written as

$$i(m\theta-nB\Omega t)$$

$$L_{mn}(x,r,\theta,t) = L_{mn}(x,r) e , (30)$$

where

$$m = nB + kV , \qquad (31)$$

and where k can assume arbitrary integral values (positive, negative or zero).

The circumferential phase velocity c $(r)_{mn}$ associated with Eqs. 30 and 31 can be written as

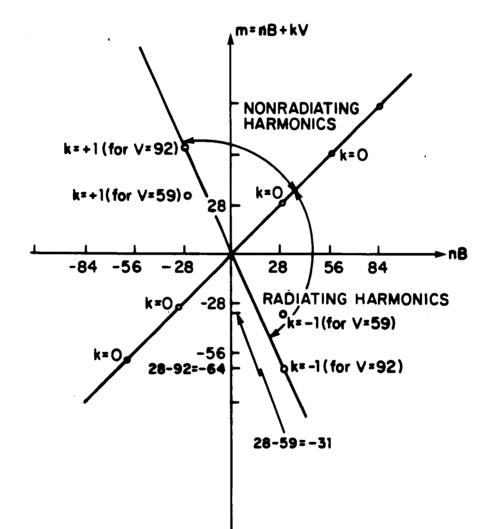
$$c_{o}(r)_{mn} = \frac{nB\Omega r}{nB+kV} , \qquad (32)$$

Similarly, from Eq. 19, the circumferential phase velocity for *all* the rotor harmonics n is Ωr and assumes the value Ωr_t at the SBLE tips. This same value is recovered for the interaction modes from Eq. 32, for the stator fundamental mode, i.e., for the case k = 0.

Figure 44 depicts the situation in terms of these rotorstator interaction harmonics. The harmonics lie at the intersections of vertical lines passing through the nB axis for $n = 0, \pm 1, \pm 2$... and horizontal lines passing through the m axis for $k = 0, \pm 1, \pm 2$... The rotor fundamental tone occurring at the blade passage frequency f_r (see Eqs. 17 and 18.) corresponds to vertical straight lines passing through $n = \pm 1$ (i.e., $nB = \pm 28$). Similarly, nth rotor harmonic corresponds to frequency nf_r . The fact that attention was turned to the stator fundamental harmonic at frequency fr (see Eqs. 19 and 20) means that the k = 0 stator mode was examined at f_m . From Eq. 23 , one finds that this stator fundamental harmonic barely escapes radiation. From Eq. 32, it can be seen that the same situation applies to stator fundamental harmonics (i.e., k = 0 modes) for all rotor harmonics (i.e., arbitrary n). Thus, the straight line in Fig. 44 passing through these k = 0 modes separates the radiating and non-radiating harmonics.

The criterion of Eq. (24) was applied to prevent the next candidate stator harmonics $(k = \pm 1 \mod 5 \ n = \pm 1)$ from radiating at the blade passage frequency f (only). The straight line joining these k = -1, $\pm 1 \mod 5$ thus also separates the radiating and non-radiating harmonics. The flow-induced assymmetry in radiation span along wavenumber is reflected in Fig. 44 by assymmetry of radiating and non-radiating harmonics around m and nB axes. Incidentally, stator harmonics lying in the upper right and lower left quadrants of the m, nB plane possess circumferential velocities that are oriented in the same direction as shaft rotation Ω , and the harmonics lying in the upper left and lower right quadrants possess velocities that are oriented in the direction opposed to Ω .

Finally, note that the relatively high number V of stator blades indicated by Eq. 28 may cause design problems of aerodynamic nature. For example, relatively high solidity at the hub, particularly for the scale model fan, is unacepceptable. Therefore, a compromise number of 59 was selected for V. Such a choice ensures circumferential cancellation at the SBLE hub, but not at the tip. In other words, with reference to Fig. 44, $k = \pm 1$ modes would radiate from the SBLE tips.



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FIG. 44. SKETCH OF RADIATING AND NON-RADIATING ROTOR/STATOR INTERACTION OF HARMONICS (nB,m).

6.3 Stator Aerodynamic Design Considerations

The stator is characterized by a backward leading edge sweep varying from 25 to 40° in the meridional plane. The axial spacing between rotor trailing edge (TE) and stator leading edge (LE) varies approximately from two to three rotor hub chords along the span.

A minimum number of 59 blades was specified by acoustic considerations. This, in conjunction with a tentatively selected hub cascade solidity $\sigma_{\rm hub}$ = 2, resulted in a chord length of approximately 30mm.

The meridional contour of the stator was shown in Fig. 14. Radial station 17 crosses the leading edge, 19 the trailing edge, and 18 crosses both the leading and trailing edges. Radial equilibrium along those stations is markedly influenced by the varying degree of stator turning, resulting in peculiar tangential velocity distributions that have been input in R-121, together with the corresponding total pressure loss distributions. The flow conditions from rotor exit station 13 to stator inlet stations 16, 17, 18, and 19, have been calculated according to constant rotor exit momentum V \cdot r specified along the streamlines.

The meridional flow pattern (Fig. 14) shows the radial streamline shifts induced by the swept back stator configuration, especially in the hub region, where $V_u \cdot r$ is large and has a strong effect on radial equilibrium. In the axisymmetric flow case, the streamlines approaching the leading edge are deflected inboard. Looking at the lower portion of station 17, the flow at the hub section has already undergone the major part of its turning, and $V_u \cdot r$ thus increases markedly from the hub to the 30% streamline on that station. This substantial departure from free-vortex flows generates an increase of the axial velocity component toward the hub and a corresponding increase of the mass flow density ρV_x , in turn resulting in inboard streamline shifts between leading edge and station 17. This characteristic pattern is found along the entire span, but disappears gradually toward the tip section because of the decreasing value of the V_u^2/r -term.

The meridional streamline curvature term V_m^2/R_c has a strong effect in this flow field region. The determination of R_c how-ever is very approximative even with the spline-on spline procedure

used in R-121, so that the interpolated values of the flow conditions at the stator leading edge cannot be expected to be smooth. Fig. 45 shows the radial distribution of the inlet angles α_3

determined by V_x -interpolation and constant $V_u \cdot r$ along the stream-

lines, and the smooth distribution assumed for blading design. The maximum smoothing error does not exceed $1-1.5^{\circ}$, which is well within the accuracy that can be expected from the axisymmetric analysis.

Figure 45 also shows the stator exit flow deviation angles δ calculated with Carter's formula [Equation (8), circular mean camber lines]. The blade sections are stacked with the leading edge in a meridional plane. All profiles were set at 0° nominal incidence. Table 6 lists the profile data defining plane sections perpendicular to the radius in the leading edge plane.

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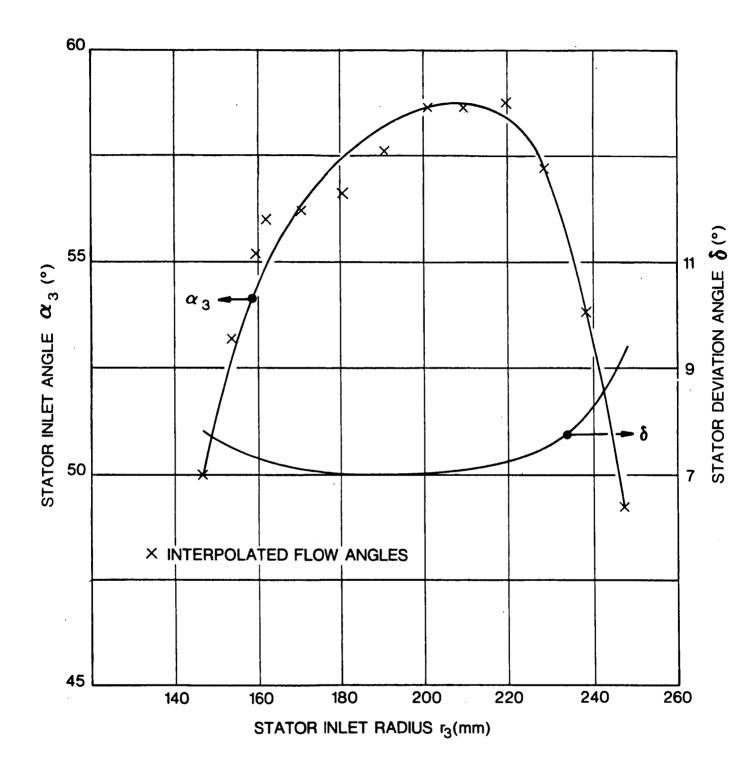


FIG. 45. STATOR INLET FLOW AND DEVIATION ANGLES.

STATOR BLADE DATA TABLE 6.

	59 Blades	es						Ld)	(Plane Sections)	tions)
Section Radius (mm)	147	154	160	171	191	210	228	236	242	248
Mean Camber Angle	47.80	44.90	43.00	44.90 43.00 40.80 38.90 38.40 40.00 43.1	38.90	38.40	40.00	43.1	46.50	50.20
Setting Angle γ (°)	73.90	75.05	05 75.80	76.70	77.55	77.90	77.50	77.50 76.55	75.45	74.30
Chord Length c (mm)	31.3	31.3	31.5	31.85	33.05	34.80	36.90	37.9	38.75	39.60
Diffusion Factor D	.396									.288
Rel. Thickness v (%)	6.0	1	1	I	1	I	I	I	I	6.0
Axial Coordinate of L.E. z _L (mm) *	0	4.1	7.8	14.0	25.1	35.0	47.4	53.7	58.7	64.0

*Leading Edge swept back in a meridional plane.

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SECTION 7

COMMENTS ON RESIDUAL NOISE SOURCES AND NOISE LEVELS OF THE SWEPT ROTOR AND STATOR FAN STAGE

The object of the Low Source Noise Fan Program is to design fan components for minimal noise generation. This has been done by first using physical models for each of the component noise mechanisms, and calculating the appropriate parameters from the particular baseline fan design, then modifying the component geometry to minimize noise generation. All sources of noise cannot be eliminated, and indeed all sources have not been attacked in this study.

7.1 Residual Sources for a Fan outside the Laboratory Environment.

As has been previously discussed in detail, the compound sweep required on the rotor blades for structural reasons will lead to a conical shock at the sweep reversal point. However, in some future fan designs, the location of the sweep reversal point at a radius less than that at which the critical relative Mach number occurs will eliminate the source of noise. Rotor discrete frequency mechanisms which cannot be eliminated include the so-called Gutin noise sources associated with steady loads and thickness noise. However, the non-radial blading may cause these mechanisms to excite high order duct modes and thus reduce the radiation to the far field. Rotor broadband mechanisms are relatively poorly understood quantitatively (in the absence of inflow turbulence), and thus are difficult to attack at Shock/turbulence interaction in the channel may cause the source. some forward radiated noise, and quite likely causes aft-radiated broadband noise.

Stator noise mechanisms are much better understood and can be attacked with much more confidence than some rotor mechanisms. The uncancelled tip radiation (calculated in Appendix C) is the only discrete-frequency mechanism inherently associated with the subsonic trace speed swept stator concept, assuming that the rotor wake field can be accurately specified. Stator broadband mechanisms not attacked by the swept leading edge include vortex shedding and flow separation at the trailing edge.

Other broadband noise from an installed fan comes from the exhaust jet and duct boundary layer turbulence interaction with the lip of the fan duct. 7.2 Prediction of Noise Levels and Noise Reduction of the Swept Rotor and Stator Fan.

Despite intensive research efforts in the past twenty years which have led to a good understanding of noise mechanisms and scaling laws, the ability to predict fan noise for an arbitrary design on a component-by-component basis is quite limited. For conventional fans, useful semi-empirical correlations of data have been made using scaling laws which are based on assumed mechanisms. Thus, for conventional fans, one can predict within a few dB the expected sound power and directivity. However, the applicability of those correlations to a fan of unconventional component design is doubtful.

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For the subject fan design, the prediction of residual noise from the rotor requires the knowledge of the strength of the conical shock upstream of the rotor, which is not presently known due to the cessation of activity on the 3-D compressible flow program. The stator discrete noise has been calculated directly for basic principles and is presented in Appendix C.

However, the main noise source of interest, rotor multiple pure tones cannot be reliably estimated without detailed information on shock structure and duct propagation characteristics. In the interest of providing an estimate of the benefits of eliminating MPT noise, a computer program published by Burdsall *et al.*, (Ref. 20) was exercised (see Appendix D for details). The results summarized below for a full scale (a 40,000 lb thrust) counterpart of the 20 inch fan built in this program, show that elimination of the shock-generated MPT's reduces the overall and perceived noise levels by 4-6 dB, and reduces the tonal content in the 1/3 octave band containing the blade passage frequency by about 10 dB.

TABLE 7.	ORDER-OF-MAGNITUDE	EMPIRICAL	ESTIMATE	OF NOISE	LEVELS
	FROM FULL SCALE SI	NGLE STAGE	FAN.		

Spectrum Component	Overall PWL (dB re 10 ⁻¹² w)	OASPL*(@150') dB(re 2x10 ⁻⁵ N/m²)	PNL*(@150')
M.P.T.(conven- tional blades)	152-154	104-106	117-119
B.P. Tone	143	95	108
Broadband Mech- anisms	150	103	114
TOTAL with MPT's without MPT's	153.5-155.2 150	105.3-107.2 103	118-120 114

*Valid in the forward-radiated direction at azimuths from 40-80° from fan axis (± 3 dB); to scale to greater distances, subtract 20 log r/150, where r is distance in feet.

SECTION 8

MECHANICAL DESIGN ASPECTS AND FACILITY INTEGRATION

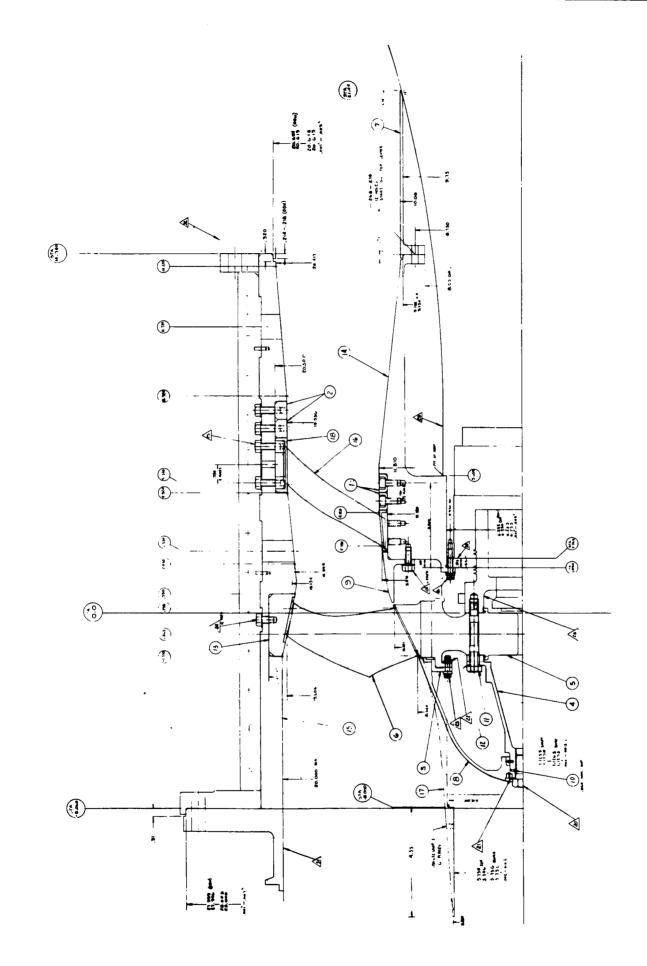
The fan rig is built to conventional standards and is designed to interface with the NASA W-2 and W-8 test facilities. The W-2, acoustic facility is arranged for the measurement of forward and rearward radiated noise; thus, the rig casings have flanges at both ends which mate with the facility mounting flanges. The manner in which this is accomplished is shown schematically in Fig. 12. In the reverse flow mode, for backward radiated noise, an additional flow path adaptor supplied by NASA and not shown in Fig. 12 is fitted to the fan outlet flanges. In the W-8 facility the fan is mounted on its rear flange with the flow entering from the bellmouth. All detailed performance measurements of the fan will be made in the W-8 facility.

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A flow path adaptor fits over the facility bearing housing to control the fan outlet flow and into this adaptor is fastened the inner shroud of the stator vane assembly. The outer shroud of the stator vane assembly is located in the fan casing and provision is made for axial adjustment of the stator by relocating the spacers at the inner and outer shrouds. The fan outer casing is split in the vertical plane for assembly purposes. The section of the outer casing in the area of the blade tips is relieved and an abradable shroud lining is installed to prevent blade tip damage in case of tip rubs. Figure 46 shows the engineering cross-section of the fan which details all the major components. Strain gauges will be applied to the rotor blades, the wires being led down the front and rear faces of the disc. In the W-2 facility the slipring is installed at the driven end of the rig shaft and, thus, the strain gauge wiring will pass down the length of the hollow shaft. When the fan is running in the W-8 aerodynamic facility, the strain gauge wiring will be led forward through the driveshaft adaptor which is fitted in place of the spinner support cone. A static fairing is installed over the slipring to provide a smooth flow profile into the fan, in place of the spinner.



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FIG. 46. DETAILED CROSS-SECTION OF FAN RIG.

SECTION 9

CONCLUDING REMARKS

A research program was undertaken to try to demonstrate that source noise reduction concepts which are based upon full and rigorous application of fundamental aeroacoustic principles can be implemented on turbofans in the currently-operating range of tip speeds and pressure ratios, utilizing the existing design and manufacturing capabilities of the aircraft engine industry, without serious compromise of the noise reduction concept.

The subsonic leading edge rotor blade concept has significant potential as a practical solution to rotor-generated noise due to its inherent lack of sensitivity to off-design-point operating conditions, and the large family of detailed edge and generating surface contours available for fans in various speed ranges. The aerodynamic behavior of subsonic leading edge rotors in supersonic absolute inflow velocities is largely unknown at this point in time. However, it is believed that the characterizations of such flow fields, to the extent necessary in developing actual engines using the subsonic leading edge rotor principle, would at this time require considerably less effort than has been expended historically in understanding aerodynamic behavior of conventional rotors.

The subsonic trace speed stator vane concept can be implemented through application of moving medium acoustic principles and a knowledge of the details of the rotor wake field, the lack of the latter being a current limitation. However, the subsonic trace speed concept can, in principle, be successfully implemented by use of conservative assumptions about the rotor wake field.

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APPENDIX A

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COMPUTER LISTING OF AEROTHERMODYNAMIC PARAMETERS FOR FINAL ROTOR, STATOR & FLOW PATH DESIGN

APPENDIX A: DETAILED AEROTHERMODYNAMIC DATA

This appendix contains a computer listing of the aerothermodynamic data for the final design of the fan.

The first four pages, A-6 to A-9 are input data to AVCO Lycoming Program R121 at various axial stations. All units in the SI system and headings on the columns are self-explanatory. The three parameters in the left hand column are:

TOT	PRESS	=	Total	Pressure Rat	t10
TOT	TEMP	=	Total	Temperature	Ratio

VU = Absolute Tangential Velocity Component of the Air (m/sec)

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The remaining pages are detailed output at the various axial stations, the non-obvious terms of which are defined below.

Coded Term	Meaning	Units
A STATIC	ambient sound speed	m/sec
A TOTAL	sound speed based on total temperature	
ALPHA BAR	<pre>sin⁻¹ (Vm/V)= angle of flow made by V in tangential direction measured on a cone</pre>	degrees
ALPHA	<pre>sin⁻¹ (Vx/V) = angle made by projec- tion of absolute velocity vector (on a cylinder)</pre>	degrees
BETA	angle that the relative velocity vector makes with a cylinder	degrees
v	absolute velocity	m/s
VM	meridional component of V	m/s
VR	radial component of V	m/s
S-VALUE	radial length measured along a station cut (origin at hub)	m
% SPAN	percent radial distance compared to full span measured from hub	
VX	axial component of absolute velocity	m/s
VU	tangential component of absolute velocity	m/s
W	relative velocity	m/s
WU	tangential component of relative velocity	m/s
MV	Mach number of absolute velocity V	
MVX	Mach number of VX	
MVM	Mach number of VM	

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Coded Term	Meaning	Units
R-ADC	streamline radius of curvature in meridional plane	m
RHO	fluid density	Kg/m ³
ROTOR EFF	polytropic rotor efficiency	
S-VALUE	radial length measured along a station cut (origin at hub)	m
STAT PRESS	static pressure	bars
STAT TEMP	static temperature	° Kelvin
то/то (то/то)т }	Similar definitions for the stagnation temperatures.	
TOT PRESS	total pressure ratio	
TOT TEMP	total temperature	° Kelvin
U	rotational speed	m/s
v	absolute velocity	m/s
VM	meridional component of V	m/s
VR	radial component of V	m/s
VU	tangential component of absolute velocity	m/s
VX	axial component of absolute velocity	m/s
W	relative velocity	m/s
WU	tangential component of relative velocity	m/s
X-VALUE	axial location of station re:origin (station 4)	m

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Coded Term	Meaning	Units
% AREA	% of annulus area taken up by a stream tube from the preceding area to that where % AREA is indicated	. %

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MASS FLOW	0007 . IF	3 1.01 289.00 0.0	א אין דורא מ	32.2000	1.461 335.70	MASS FLINW RATE	32.2000	1.661 335.70 0.0	MACS FI. TH DATE	32.2000	1.441 335.70 0.0	MASS FINW PATE	37.7000	137.00 137.00 0.0	MASS ELNW RATE	37.7000	1.44.1 737.00 113.30	
er L.	000	1.01 2.84.00 0.0	a L	000	l 1.661 334.80	Cr U	00	1.661 334.80 0.0	5	ÛŬ	1.461 334.80 0.0		οu	1.661 335.70 0.0	ĩ	00	1.661 335.70 115.80	
STATION 12 MACH NIJMREI	0*2000	1.01 288.00 0.0	STATION 13 Mach Niimre	0 . R000	1 1.661 334.60	STATION 14 Mach Nijmrer	0.4001	1.561 334.60 0.0	STATION 15 Mach Niimrer	0°400	1 1.661 334.60 0.0	TATION 16 Mach nimmer	0.5000	1.661 335.00 0.0	STATION 17 Mach Nhimrer	0.5000	1.661 335.00 120.30	STATION 19
T TO AXIAL S	0.0430	3 1.01 288.00 0.0	T TU AXTAL S	0.0715	1 1.661 334.60	T TO AXIAL S	00-0-0	1 1.661 334.60 0.0	AL S	150	1.461 334.60 0.0	AL S	0.1350	1.661 334.80 0.0	AL S	.550	1.661 334.80 126.00	axtat st
TT TUPUT TU TTP STA	••	1.01 288.00 0.0	INDUT TU TIP STA	••	1 1.661 334.60	1 NPUT TO TIP STA	••0	1.1.661 1.660 0.0	iveijt tn AXI TIP stati <u>f</u> n	0.11	1 1.661 334.60 0.0	I NOLLT TO AXI	0.1	1.661 334.90 0.0	NULT TO AXI	0.1	1.461 334.90 132.60	UI INGN.
PADI US	0•2490	3 1.013 288.00 0.0	PADTUS	0.2430	1 1.661 334.70	RANTUS	0.2400	1 1.661 334.70 0.0	RANTUS	0.2395	1 1.661 334.70 0.0	Ś	0.2420	1 1.661 335.00 0.0	1 ADTUS	2440	1.654 335.00 117.70	-
a it	Ó	1.013 288.00 0.0		o	1 1.661 334.70	TIP &	Ó	1 1.661 334.70 0.0	tlo q	ŏ	1 1.661 734.70 0.0	TIP RAPIU	č	1 1.661 335.10 0.0	TIP RANIJS	• •	1.649 335.10 107.20	
NDI TATZ	0•0200	1.01 288.00 0.0	STATION	0.0700	1 1.661 335.20	STATION	0-00-0	1 1.661 335.20 0.0	STATION	0.1150	1 1.661 335.20 0.0	HUR STATION	0.1350	1 1.661 335.30 0.0	STATION	0.1550	3 1.645 335.30 91.90	
HIB S		1.01 289.00 0.0	S BUH		1 1.661 336.30	S BITH	•	1.65 336.30 0.0	HUP S'	•	1 1.661 336.30 0.0	HUR S1	U	1 1.661 335.50 0.0	HUR S1	U	1.638 335.50 76.30	I.
RADIUS	0.1100	288.00	RADIUS	0.1340	5 1.661 339.10	RADIUS	0.1420	338.10 338.10 0.0	2 AD LUS	0.1450	338.10 0.0	RADIUS	0.1470	335.70 235.70 0.0	ADTUS	0-1490	1.634 3 <u>35.7</u> 0 60.00	
HUR		TOT PRESS TJT TE40 VU	HUH	•	TOT PRESS	вСн		TŨT PRESS TJT TEMP VU	HIB	•	<u>IDT PRESS</u> TOT TENP VU	u CH		TOT PRESS TOT TE4P VU	WUH		TOT PRESS TOT TEMP VU	
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			18450.0		0007.55	~	0.5001	000	0• 2400	2480	0•2	2900	•	0.1500	-
			200	0 Å T C	ss einu	A A	STATION 23 Mach Niimrer	Š	INDUT TO AXIAL TIP STATION	s	TIP RADIU	STATION	HIIB STA	SUIDE	HUB
0 <u>1.620</u> 342.40 0.0	341.40 0.0	1.620 340.50 0.0	1.420 339.60 0.0	1.620 338.70 0.0	1.620 337.00 0.0	1.620 335.70 0.0	1.620 335.00 0.0	1.620 334.80 0.0	1.670 334.00 0.0	1.620 335.00 0.0	1.620 335.10 0.0	1.620 335.30 0.0	1.620 335.50 0.0	335.70 335.70 0.0	TENP
			19450.0		32.7000	0	0-5000	00	0.2600	0.2480	0•2	• 2600	ò	0.1500	
			7 2 7	₽ÅŦE	SS FLOW	4	STATION 22 Mach Numrer	۲ ک	INPUJT TU AXI TIP STATION	RADTHS	TIP RAD	STATION	HUR STI	RADIUS	BUH
0 1.620 347.40	1.620 341.40 0.0	1.620 340.50 0.0	1.670 339.60 0.0	1.420 734.70 0.0	1.220 0.02 0.00	1.670 335.70 0.0	335.00 335.00 0.0	1.6?0 334.80 0.0	1.620 334.90 0.0	1.620 335.00 0.0	1.620 335.10 0.0	1.620 335.30 0.0	1.620 335.50 0.0	1.620 335.70 0.0	TOT PRESS TOT TEMP VU
			18450.O		32.7000	0	0.500	00	0.24	2480	•	2400		0.1	
		•	X G G	PA⊤E	155 FLAW	ср МА	STATION 21 Mach Numbe	AL	NPUT TO AXI TTP STATION	N I	TTP PADIN	STATION	HUB ST	SUTUS	HJR.
8 1.641 342.40 62.60	1.638 341.40 53.00	1.634 340.50 39.80	1.620 339.60 27.20	1.425 738.70 15.00	0.0 0.0 0.0	1.620 335.70 0.0	1.620 335.00 0.0	1.670 734.80 0.0	1.620 34.00 0.0	1.620 335.00 0.0	335.10 335.10 0.0	1.620 335.30 0.0	335.50 335.50 0.0	1.620 335.70 0.0	TAT PRESS TOT TEMP VU
] 8450.0	_	32.7000	ç	0.5000	150	0.2	2490	0	.2150	c	0.1500	
	1 1 1		Wda	P A T F	ASS FLOW	A M	STATION 20 Mach Niimber	AL	NPIJT TŲ TIP STAT	RADTUS	TIP RA	STATION	HUR ST	SUIDE	HUR
1 1.661 342.40 125.20	341.40 341.40 122.10	1.658 340.50 115.30	101.30	338.70 87.80	1.543 337.00 67.40	1.635 335.70 47.50	1.527 335.00 21.50	1.620 334.80 0.0	1.620 34.90 0.0	1.620 335.00 0.0	1.620 335.10 0.0	1.620 335.30 0.0	335.50 30.0		TDT PRESS TJT TEMP VU
			18450.0	C	32.7000	6	0.5000	9=0	0.1	2480	•0	0.1950	0	0.1500	
	i		A A	RATE	MASS FLOW		TATION 19 Mach Number	د د	INPUT TU AXIA TIP STATION	T	TIP RA	STATION	HUB ST	s ad l us	ния
61 1.661 0 342.40 0 126.10	1.661 341.40 123.00	1 1.661 340.50 120.20	1.66 339.60 117.50	1.661 338.70 115.20	1.561 337,00 112,50	1.661 335.70 114.80	1.654 335.00 99.60	1.646 334.80 79.50	1.636 334.90 54.00	1.627 335,00 23.60	1.621 335.10 8.00	1.620 325,30 0.0	1.620 335.50 0.0	31.620 335.70 0.0	TOT PRESS Tot Ieyp. Vu
			.18450+0	, 1 0	32.700	0	0-5000	750	0.1	2460	•0	.1750	0	0051-0	i
			Maia	RATE	MASS FLOW	Cr	MACH NIJMRF	TATION M	TIP STAT	RANTUS	TJP RA	STAT ICN	HIJA ST	R AD TUS	HUB

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	**		1.3662596	2. 1015692	7.4519930	2. 4577154	2 5070333	2.7364960	1.9734584	1.4789704 ···	1.0799446	590167	5128	334824	0.1641465	c • c	772	0.4278703	0.4278368	0.4277867	1727242 Q	0.4277198	0.4776019	0.4276863	0.4276075	770R	0.4777142	27725	0.4277253	427731	42773	0.4777310	CHa	1.1203514	1.72021.1	00505	1.1204090	.120421.	۳.	1.1204367	-	٦.	1.1294243	-		۳.	1.1274166		
i		142445	142.07168	142,95544	147.04293	142.93301	142.92493	142.02305	142.9266A	142.03030	142.93205	0	142.93567	4150	9750.	142.03744	37	0.51627		0.68836	0.75990	N.87578	0* 94 253	1.04575		2		٠	٠	•	02.0	1.57055	RAJF	000000.56250	000000.56250	000000.54250	000000.56250	1.024331000000.56250	1.005071000000.56250	000000.56250	.751051000000.56250	97841000000.56250	.43791100000.567F0	0.280261000000.56259	0.705591000000.56250	.134211000000.54750	R01000000.56250	000000.56250	
:	, 200 200	47 885 44	142.07148	42°0554	¢ 2•	~	2.0249	020000	2.0266	202	ñ.	147.03567		~	47fo.	142.03744	X > 7	0.42787	427R	0.42774	0.477KR	1,42745	n.42763	0.42763	0.42765	0.47760	4	0.42772	0.4777	•	4.	622670	7 d 1	0*0 100	547541	0.8423710(0.0878810(1.0253210(1.00507100	0°8046710	0.7510510	0.50784100	01106570	0.7807610	0.70559100	0.1242110	n.n6" R010	c.	
	BETA BAR	24*04436	1315.32161		145.74355	148.78342	153.01431	155 R8402	157 . 00532	159,63330	160.05241	162.04397	162.52395	162.96698	3.3792	153.76009	N _M	0-42787	•	0.42779	••		0.42769	0.42760	0.42770	٦.	12263.0	£7773	•	0.42773	0.42773	0.42773	APEA	0.0	4°40804	4.90032	4°00080	5 • 000 2	10.01045	•	10.00064	10.00022	0	0,00050	4,00005	4,00073	4 ,90950	5.00001	
	BETA	124.04435	135.32292	141.58029	145°74748	148.78746	153.01785		10700.721	150,63479	169.95299	5	162.52396	\$	163.37922	63 . 760°	j	-96,60394	-144.58590	•	-200.88014	-235,95805	-280.67944	-110,26978	-353,67310	-385.01343	-413.99755	-441.06079	-453,99297	-466.56616	-478.9 <u>009</u>	-490.74780	TOT TEMP		α	7 AR. 00000	7 RR. 00000	2 R.8. 00000	288.00000	2 P.8.00000	288.00000	7AR.00000	289.00000	288.0000	œ	α	2 RR. 0000	æ	
	•	35353F	340.27535	340.23535	340.23535	240.23535	340.23535		340.23535	340.23535	340.23525	340.23535	<u></u>	\sim	340.23535	340*5252	3	172.55811	101255200	730.03735	753.94171	275.78900	114.07362	349. R0005	381.46118	410.68774	37. CE	463.F4231	475.06240	487.97021	400,68921	511.14038	TUT DDFCC	1.01350	1.01350	1.01350	.01	1.01350	.01	•01	1.01350	1.01350	1.01250	1.01350	1.01350	10.	.013	1.01350	
				89, 00097	89.90997	89,99097	89, 90097	90°00097	89° 00097	89,00007	89° cộc 47	89,00097	89. 99097	89. 09047	89,00007	80, 99197			0.0		0.0	0-0	0.0		0.0	0.0	0.0	0.0	0.0	0.0	0.0	0•0	CTAT TEND	277.8	277. 82837	277, 83081	277. 83252	277. 83374	277.83496	277. P3521	277.83472	277.83423	277. 8339R	277.83350	277.83350	277. 83375	277.83225	277, 83225	
	ALPHA BAR	R9.09057	89,00057	89.90097	R0.000C7	89° 00007	Ko aara7	89,00057	80.90007	90,00097	Pc 09007	80,09097	P9 ,90007	80,000 97	90,00097	89,90097	~~	147 68244		147.93005	147.071 80	142.01103	142.90294	142.90555	142.91440	142.07264	142.00708	147.03306	4	147.03704	2FT EP. 541	142.03744	6717 DDECC	49505	0.89366	0.89368	0.89370	0.89377	0.89373	0. 89373	0.89372	0.89372	0. 89372	r. 80271	0.89371	0.89371	17508.0	0.89371	
	A STATIC	334.17236	334.17334	334°17456	334.17578	334.17651	334.17725	334.17725	334.17700	334.17676	334.17651	334.17627	234.17677	334.17603	334.17603	734.17603			0.0	1 21402	28.74154	35.33066	44.70258	56.49356	65.27221	73.17368	80.52478	87.39366	90.67471	93.86475	56.97093	100.00000	=	96.60364	144.58589	180.21828	209 88914	235.85805	280.67944	319.26578	353.67310	385.01347	413.99755	441.06079	453.99202	466.56616	478.80908	490.747R0	
	RADIUS	0.05000	0.07483	0.09328		0.12207			0-18205	0.19077	0.21427	0.22828	0.73408	0.24148	0.24782	0.75400		S-VALUT D. D				0.07207	0-00527	11525	0.13205	0-14977	3.16427	0.17828	0.18404	0.19148	0.19782	0.20400			-0.76000	-0-26000	-0-26000	-0-26090	-0-26000	-0.26000	-0.26000	-0-26000	-0-26000	-0.76000	-0.26000	-0.26000	-0.26030	-0.26000	

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Ua/ua	1.0000	1. 00000	1.00000	1.00000	1.00000	1.00000	1.00700	1.0000	1.00000	00000	1-0000	1.0000	00000		1.0000
19-4791/4A	0.02244	0.01692	0.00976	0.00556	0.00.00	0.00062	-0.00037	-0°001E	-0.00087	-0,000 AB	-0.00082	-0-00079	-0.00076	-0.00070	-0.00045
(ad)/du	96550.0	0.03584	0.02713	0.01032	0.01290	0.0035R	-0.00274	-0°0065 P	-0°00cai	-0.011 PR	-0.1305	-0-01343	7 3 5 10 0 -		-0.01344
ET 4P	0.0	0*0	0.0	0.0	0.0	0.0	0.0	0•0	0.0						0.0
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A STATIC	236,35059	734 . 09341	334.58506	334.49365	334,35815	14.16333	334.04107	222.05077	223 00281	ACAS READE		100.000.000	01/Cu*ccc	3 3 3 6 9 2 6 2 6 3 3 9 3 6 9 2 6 2 6 2 6 2 6 2 6 2 6 2 6 2 6 2 6	333. 82050	NY CO .		12.20845	22.42175	79.98386	36.5367	47.78571	57.41283	65.96944	73.75185	80. 941P5	87.66CP3	00.87053	93.9926	57.0323	100.00000	2	96.99026	144.97260	185.25151	215.0287A	740.82565	285.12769	323,03531	356.73535	387.38281	415.65702	442.15674	454° 79639	467.0ACP4	479.06454	490.74780
SILLUN	0.05020	0.37712	0.00400	0.11131	0.12466	0.14750	1.16721	0.19465		0.21616	01617°0		75155 C	0/14/°r	0.455			0.00	0.04570		0-07666	0.70720		0_13445	0.15021		0-17865	0.18519		0.19775	0.20280	Y-VALUE	-0.17000	-0-17000	-0.17000	-0.17000	-0.17000	-0.17000	-0.17000	-0.17000	-0.1	-0.17000	-0-1	-0.17000	-0.17000	ł	-0.17000

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ВЕТА ВАР 122.15840 120.0111	1 39 05345 1 43 80153	147.04417	140.46158	152.01524	155.37073	157.24905	150.75227	150.99771	161.05333	161.52483	161.94402	2.3769	142.74417	41	0.34262	0.41051	0.41500	0.42540	CO2F4.0	0.44144	0.4471¤	0.45080	766376	707570	<u>0</u> 45505	0.45427	0.45650	0.45665	1.45672	ድ ልቦርሏ	0.0	E.67079	51155.3	5 <u>1</u> 076ª	5,10AQ4	10.08200	0.04611	0 . REJER	5001s	9.775J7	a.75368	4.87046	4.RAGO	4.84643	4.84570
RETA 132.43042 132.37237	1697° 06470	147.14690	140.53735		1 EE. JOE60	157.26215	158.75075		161.05470	161.5554	161.94524	142.37686	152.76419	5	-104.3228	-1 54.6459	-100.12096	7F715.0JC-	-244.62959	-724.14868	-325.42578	-258,58780	-388.77441	-4]6.68284	-442.78747	-455.25489	-467.39867	-479,71143	-40 0. 747RJ	THT TEMP	284.00000	284.00000	288.00000	00000°8ac	7 R.R. 0000.	7 PA .00000	2 88.00000	788.0000	2PR.00000	7 P.R. 00000	798.0000	788.00000	2 8 8 0 0 0 0 0 0 C	788.00000	200 0000
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AL PHA 89. cora7	89, coro7 89, coro7	80, 000 3	89.99507	Re, grag7	89. 10051	000	80°0000	80,00097	89 , 00007	ga, agra7	89. gccg7	Ac. ccnq7	ka cci a 1	ī	0.0	C • C	0.0						0.0	0.0	0.0	0•0	0.0	0.0	0*0	STAT TEVD	281.29258	279.04736	278.35548	277.03904	277.63574	277.10487	276.92407	276.7465R	275.£2866	276.55176	276.50317	276.48706	276.47607	276.46875	776 A4660
АŁРНА РАR Rn_ 90007	89,00007 80,00007	0000	Ro. acca7	89,00057	U.	0005	45,000,04		80°00003	89 ac on 7	80, 50017	ຊດູດາ <u>ດ</u> ດ7	Rn.cace7	λΛ	, , .	75751.551	1 39.12.42	141 67875	143.88775	147.04171	140, (2000	150.27897	151.1C965	151. FACCO	151,08041	152.10027	52.1	52.2278	1,52。24841	STAT PRESS	0.03443	501	0. Aca72	\$0a.	. Roi4	. R965	0.09257	α.	<u>د</u>	15079.0	799	а. •	795	- ÷	0 7 0 /
4 STATIC 736.31006	234 . 90552 224 40465	234.24023	334.05737	233.79199	333.62851	333.52197	333.45053	733.40454	333.37524	233.36572	373.35PRO	333°35440	333.35254	N C D A N		13.02875	CE112.20						72.61345	80.83517	87 . 58826	90.81572	03°95547	97.01474	100.00000	=	104°3322ª	154.646 ^{c g}	190.1295A	219.31737	244.62950	2 98.1486 8	325.42579	358.58789	388.77441	416.68384	442.78247	455 . <u>2</u> 5489	67.3RA6	479.21143	00 7670
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VW 124.60940 132.47052	┍┥┍╵┍┥┍	466.01677 48.07009 50.31824 51.83570 51.17830 52.28340			0.45405 0.4564295 0.45642985 0.45642985 0.4570767 0.4571050 0.4570737	РНП 1.1459634 1.1459634 1.1275944 1.1275944 1.1278938 1.195978 1.1145394	1.11/6037 1.1005005 1.1003315 1.1073335 1.1073333 1.1073333 1.1074265 1.1065305 1.1066302
		46° 91670 48° 07009 50° 31824 51° 81570 51° 81570 52° 17820 52° 28340	4691 71990 6700 1127 1127 1127 1127	- F A' F- A B A			رأيتو أيتو أيتو أيتو أيتو التواقيو التوا
V 60040 67052			152.3260 152.3710 152.3710 152.3710 0.5112 0.5712 0.5770	0, 84275 0, 84275 0, 976275 1, 08167 1, 1765 1, 27518 1, 37385	1.40,41 1.40,41 1.445,82 1.475,82 1.5005 1.5005	RANC 0.44054 0.46694 0.67758 0.7781 9 0.7781 9 1.33733	1.81611 2.45787 3.97551 3.97551 4.01944 4.201945 1.2.70243 1.2.70243 25.00371 277,55763
124]37.00711 141.20777 142.55595 146.01470	140.0100 140.0100 151.01924 151.02111 151.81670 152.17830 152.17830 152.17830	152,34601 152,37100 152,37100 152,37700 15,37700 15,37700 15,37746 0,35746 0,35746 0,40746		0.4544 0.4549 0.45701 0.45701 0.45701 0.45701	1 6 705 1 6 8005 1 0 74876 1 0 74876 1 0 7497 1 0 7500 1 0 7150 4 77331	4 4 4 4 4 4 4 4 4 4 4 4 4 4
нета Рас 133.39412 140.07743 166.067743	145.05707 147.94281 150.13729 153.31348	1,2,2,1,2,4,4,1,2,2,4,4,1,2,4,4,1,2,4,4,1,2,4,4,4,1,2,4,4,4,4	161.000 162.36751 162.35751 162.7525 162.7525 162.755 162.00 151210 151200 151210 151200 151200 151200	0.0000 0.000000	0.4550 0.45583 0.45703 0.45703 0.45711	7. 4854 0.0 5.65641 5.30069 5.23059 5.13303 10.17839	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
HE-13 34.4807 41.4726 65.2260	145.36488 148.15138 150.28011 153.30750	155.55045 155.45045 158.44077 160.04405 161.06737 161.52838	J61.06077 167.36774 167.75203 167.75203 -167.45583 -167.45583 -197.45552	-240 07734 -240 07734 -291 07734 -328 64387 -361 05205 -360 61014	-443,59607 -455,84700 -467,17246 -470,3071 -400,74781	717 7540 284,0000 284,0000 388,0000 288,0000 288,0000 288,0000	288,0000 288,0000 288,0000 288,0000 288,0000 288,00000 288,00000 288,00000 288,00000
с 11111 340 • 23535 340 • 23535 340 • 23535	440°23535 340°23535 240°23535 240°23535 240°23535	2440 2440 2440 2440 2440 2440 2440 2440	340.23535 340.23535 340.23535 340.23535 40.23535 171.4535 171.5555 246.25555 246.25555		468.c7363 480.61157 491.95681 503.03003 513.85547	T(T DDE 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350	1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350
ALTHA 80,00007 80,00007 80,00007	80° 00007 80° 00007 80° 0007 80° 0007	80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,0000 80,000000 80,000000 80,000000 80,00000000	84, 99097 84, 99097 80, 69097 90, 0 0, 0 0, 0 0, 0 0, 0		000000000000000000000000000000000000000	5741 TEW 290.26721 279.26758 278.53687 278.07788 277.74512 277.25528	276.5703 276.5516 276.5101 276.45061 276.45061 276.46081 276.46078 276.46078
АС, 37007 80, 37007 80, 90007 80, 90007 80, 00007		70000 70000 70000 70000 70000 70000 800 8	R9.0007 R9.0007 R9.0007 R9.0007 R9.0007 110.07666 136.22077 126.22077	147,65045 146,41141 148,65930 150,14725 151,12573 151,76877	152.1501 152.27713 152.34745 157.37789 152.36709	574 00555 0.01135 0.01265 0.00166 0.80264 0.88777 0.88777	0.8838 0.88164 0.88164 0.88714 0.87814 0.87835 0.87835 0.87835 0.87835
4 5,4 1(335,63354 235,03760 334,59612	234.5512 334.32324 234.12305 333.83081		223.34351 223.34131 233.34225 7 5 PAN 0.0 12.23555 21.235155 28.87123	56.53507 65.23507 56.53507 65.22737 13.15240 80.48900	87.357?5 90.64209 63.83946 96.95669 100.9000	11 7.85683 117.85683 163.45581 197.36252 225.47882 249.97734 292.29247	328.66380 361.66226 361.606226 417.98145 443.55662 443.55662 4455.84760 465.84760 467.37246
0.06100 0.07462 0.10217	0.1021/ 0.11672 0.12040 0.15130	0.1701) 0.14640 0.20218 0.21634 0.221634	0.2471 0.24413 0.24413 5-VALUF 0.0 0.02447 0.05477	0.06840 0.09070 0.10911 0.12589 0.14118 0.14118	0.16869 0.17656 0.18113 0.18713 1.18733 0.19300	x-VALUE -0.11300 -0.11300 -0.11000 -0.11000 -0.11330	

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DP/(PTP-P) -0.08872 0.00964 0.00550 0.00349	0.00228 0.00098 0.00039	0,0000 -0,00006 -0,00014	-0.00015	-0-0001- -0-00010- -0-0000-										
09/(07-0) -0.16534 0.02367 0.01706 0.01312	0.010C7 0.00568 0.00244	0.00091 -0.00067 -0.00172	-0.00222	-0.00709 -0.00179 -0.00142		·								
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БАМИА 1.47002 1.47002 1.47002 1.40002	1 • 40002 1 • 40002 1 • 40002	1.40002 1.40002 1.40002	1.40002	1.40002 1.40002 1.40002			·	·			, •			
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STATION PUJMBER 7 DOWNSTREAM OF REMOTE

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٩٧	43,8096619	33.6834106	27. 4072882	23.8422394	20.695877]	15.8085357	12.2691612	9° 3464365	6.0097004	4.820R046	2,0910392	2.1565123	1.3730574	0.6420782	-0.0261401	W/W	0. 2954141	0.4125286	0.4264694	1.4362247	0.4434553	n.45200]8	P.4602858	n. 4642775	n. 4667419	0.468714B	0.4600897	0.4693007	0.44099272	0.4701356	UHa	1.1340730	1.1273499	1,120003	1.1164608	1.1120476	7.1090437	1.1040442	1.1079902	1.1017790	1.1010523	1.1006193	1.1004658	1. • 1 0073AO	1.1002147	1,1001015
۸N	132.47841	138.02740	142.52775	145.67274	147,99673	151.34656	153.38936	154.66493	155.451.20	155.92160	156.20059	156.20945	156. 39213	156.45871	156,53415	Ţ	0.56796	0.66 0RD	0.753R3	0.45572	ŋ, aros 2	1.00184	1.0001	1.19792	1.24977	1. 34420	1.41575	1 2034 - 1	1.51522	1.54708	0 8 N C	0.60553	0.58773	0 . 60530	77228.O	n, op or R	1.30452	200001	2.03745	4.55057	7.36474	11.71530	52.727na	13.703RG	12. 54094	11.52038
>	122.47841	130.02740	142.53075	145.67774	147.99673	151.34656	152,38936	1 54.66483	154,451 20	155.02160	156.20059	154,70045	156.38712	154.45031	156.53415	X / M	712720	0.4000	0.41872	1.43024	010570	0.45130	0.45881	0.46343	0.4KK7R	1.46700	0.46900	1.44025 0.44525	0.46080	n.47014	502	10,21094	14.12477	1] ₂ 28577	80017°0	8, N3RKN	6.029RG	4 5 87 87	3.46451	2.54760	1.77176	1 - 197 - 1	0.70055	0.50207	0. 23513	-0,00057
BETA RAD	135.87700	141.98268	145.54672	148.10968	150.09705	153.05930	155.75674	156.99155	158.41568	159.61528	160.64291	161.10316	161.53250	161.92365	167.37892	2	0.39541	0.41253	0 . 47647	0.43622	0.44346	Usest"Ú	0.46079	0.46478	0.46674	0.46871	0.46909	()*D4**0	0.46000	0.47014	7 AREA	0.0	5.63661	5.30707	5.24658	5821.	10.10791	66540°6	944631	9.7A669	0,050,0	00066.0	4 . R5790	4.85535	45534	4.95144
PETA	137.53284	142.83200	146.06674	148.45753	150.34097	153.18758	155.32657	15020.1721	158.43507	150.62424	160.64619	161.10486	161.53325	97550.121	3 62 . 30882	17	-136.50709	-176.55721	-707.7465R	12121.452-	-257.34390	-297.7974K	 	-364.21R51	-307,03994	-410,60352	-444.61465		-470 63867	-490.747AD	TUT TE40	2 8 8 0000 U	2 9A.00000	244.00000	7 8 8 00000	284,00010	288.0000	2 RR. 00000	288.00000	288.00000	2 R8.0000	7 R.R. 0000	788,0000A	7 R8. CO000	œ	7 R4.0000
A TOTAL	340.23535	340.23535	340.23535	340,73535	340.23535	340.23535	340.23535	36355 .025	340.23535	340.23535	340.23525	3632560055	340, 73535	340.73535	340.23535	з	100.28902	524.10713	752.02077	775.74146	206. R6400	224.04925	366.47583	305, 69727	422.5717R	447.63672	471.25806	47.00°444	504 . 51106	515.1CR15	TUT DRFSS	1.01350	1.013=0	1.01350	1.01350	1.01250	1.01350	1.01350	1.01350	1.01250	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350
ALPHA	99, aocq7	89, 00007	40° 10002	A9.00097	80°0001	99.00097	Ro. cocq7	89 <u></u> caaa7	89°00557	89.00057	89. 95597	89.90007	o	89.00007	8ª, accc 7	117	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0	0°0			0.0	0.0	STAT TEMP	279.74660	278.5197A	277.89111	277.44043	277.JO083	276.60191	276.29199	276.09668	775.07534	275.00234	275. 05013	275. 84375	275,83081	275.Plars	75. AC71
ALPHA RAP	go coro 7	80.0007	RC. COCT	80°0407	80,00057	8c° 4049 7	ga_cq007	89,90C07	Ageo 7	80,00007	go.cacg7	40,000,09	Rc. acra7	Rc. qacc7	8c° cdcd 7	××	125.02495	123.85427	139.77403	143.70837	146。54253	150.500	152.85789	154.38716	1 cc. 20765	155,84704	154.J7195	104-24-10 167 23410	156.456PA	F.5341	STAT PRESS	0.00005	0°0146	98404	C.89930	0.88540	0.87992	0.87648	0.8743]	0. R77C7	9.P7216	0.87168	0. 87151	0.P7137	0.87124	0.8711
A STATIC	235.03711	334.58862	234.21C54	333.03970	233 ° 73535	73454.555	333 . 24805	333.13013	732 . 05689	333 . 012c4	332.96657	332.97729	332.96973	732.06265	2220 05522	T SPAN		2008	20-05672	27.54742	34.10541	45 . 526F4	55.41707	£4.27817	77.39659	70,91417	86.07630 00 35005	- 18-5 - 18- 03 - 461 40	84538.46	100.0000	. =	136.557cA	176.55721	207.74658	224.12131	257.343cc	297.75736	332.R3CP1	364.21851	3a 2° 93c c4	419.60352	444.41865	456.59644	468.26196	479.63967	490.74780
Angus	0.07070	0.00140	0.10754	E1 121.C	0.13322	3.1541 5	0.]7229	0.18952	0.70338	0.21718	0.23013	0.23633	0.24236	0.74P25	0.25400	S-VALUE	0.0	0.020.0	0.03f P4	0.05045	0.06252	J. 08245	0.10158	0.11782	J.1 3268	0.14648	6454I.0		0.17755	0.183.0	X-VALUE	-0.0000	-0.0000	-0. JR000	-0.08000	-0.08000	-0.0R000	-0.08000	-0.0R000	-0.080JO	-0.04000	-0,08000	-0.08000	-0.08000	0.08000	-0.0P000

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エーレオレート	1.00000	1.00000	1.00000	1.00000	1.00000	C0000 • L	00000	1, 00000	1.00000	1.00000	00000°i	1.00000	1.00000	00000	1.00000
*1/TN	1.00000	1.00000	1.00000	1.00000	1.00000	000001	1.00000	1.00000	1.00001	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000
t(na/na)	1.0000	1.0000	00000.	1.0000	1.00000	00000	1.0000	1.0000	1.0000	1.0000	1.00000	1.00000	1.0000	• • • • • •	1.0000
Ud/U3	1.0000	1.00000	000-00 •	1.0000	1.00000	1.00000	1,,00000	1.00001	1.0000	1.00000	00000°i	1.00001	1.00000	00000	1.0000
(d-a_4)/dG	-0.06340	-0.03070	-0.01065	-0.01544	-0.01294	-0.00974	-0.0771	-0.006 20	-0.00505	-0.00417	-0.00352	-0.0032R	-0.00307	-0.00202	-0.00279
10-10/00	-0.12275	-0.08211	-0.06574	-0.0612P	-0-05933	-0°05816	-0.05713	-0.05561	-0-05373	-0.05191	-0-05072	-0.05060	-0-05081	-0.05146	-0.05255
FT AD	0.0	0.0	0.0	0.0	0-0	0.0	0.0	0.0	0-0	0-0	0.0	0.0	0-0	0-0	0.0
ダッチタリ	- 40002	1 40002	1.40002	1.40002	1.40002	40002	1.4 1007	1 40002	40002	1 40002	1 40002	1.40002	1.40002	1.40002	1.40002

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VR 49.0544525	30,3134155	27. 21.454093	76 2216107		14.032487	10.5048742	7.6220007	5. 74506AR	3-7460477	7 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1 5AR3260	0.8234487	0.0845216			0. 4707643	0-4407420	0.4494257	0.4541809	0.4460A67	0.472253	0.475071A	0.4781604	0. 47020 <u>04</u>	0.4900166	0.4891507	0-4901606	0.4796390		CH a	1.1267309	1.1197157	1.143722	0801011.I		1 0000401	1 0071005	1.0061037	1 . DOSARK	1.0051710	1.0951042	1.005004	1.0051748	1.0053617
VW 1 28. 97268	143.42525	147.12531	157.07620	155.24237	157.20354	158.30251	159,09012	159.48146	150.68053	150. 72227	150.77644	150,67899	150.56055			C 10402 0	0. TR77R	0. 85074	1110.0	1.01944	1.11446	1.20021	1.27017	1.352R3	1.47233	4557	I 68831	5512	•	Juba	0. 44775	5.0	0.45979 22210		2 - 22 - 73 1 - 54575	2.26700	2.55.700	5.94741	10.60195	74.74722	03.22456	-34.76683	-11.37441	
V 138.07348	147 42535	14.41.41	152.07620	155.24237	157.20354	1 58, 20251	150,00013	1 FO 4 81 46	1 59.68053	150, 7237	2.1	α	1 50 , 56055	~~~~	0-20760	• •	0.42060	0.44154	75034°0	9.46770	0.47075	0.47407	0.47761	0.47013	(2001 7 ° U	0.48017	0 60001 0 60001	0.47964			21.06650	15.00PK7	x045x •> -	0404460	4 9100K	5,1211g	2,007,2	74400	1.48502	12271.1	0. PEN39	0.54976	0,79547	0.03035
11. P. C	142.44878 145 73305	1001104	49.97635	52.80515	54.93425		5 8	ŝ	160.27617	160.74110	141.17941	5	161.98868	ŇŅ	4	0.42974	0.44074	0.44043	0.45619	0.44609	0.47274	0.47597	0.47814	0.47440	0.44072	0.48015	0.4400				0.0	0-05r.c		5.12382	10.10305	0.04748	0.84844	0.70769	9 . 75988	9 . 74253	4.86644	4.R6537	4.84555	4.86753
- e v	14/2014/2014/2014/2014/2014/2014/2014/20	148.56412	150.29475	152.97021	155.7221		1527C.97231	159.75601	160.27998		141.17078	141.59786	16: "ояяба	17	-150.70709	-196.83635	-215,04522	-240°04404	-263.15322	-302.13590	-336.11604	-366.6P701		-4/0.44 -//F		-45/ . 4966 -468 41490	-4 70 - 80664	-490.747RD			ZFB.00000	788 00000		788.10001	7 R. 00000	244.00000	2 84,0000	2 R. n0000	788,00000	2 P.R. 00000	2 P.B. 00000	2 P.A. 00000	788,00000	288.00000
& TOTAL 340.23535 340.23535	340.02525	340.23525	340. 23525	340.23535	340.23525	340.23535	340.23535	340.23535	340.23525	340.23535	340.73525	53	340 . 23535	з	704.00951	735.53900	٣		303,03555		4/ 200° 1/ E			673 16676		404 V8030		516.035R0			0-610-1		01250	01350		01350						1.01350		1.01350
AL PHA A0,00007 A0,00007	80,00007	R9, 00097	89.95097	Ro. geeg7	89. 300 GT	Rq. gccg7	89,00007	89 . 99997	89 . 99097	80°00001	ð	ċ	R9, 90097	ÎIA	0•0	0.0	0.0	0.0	0.0	c•0	0.0						0.0		011 TEMD	1000 04	277 76267	277.22876	276. 81690	276.40170	276.00757	275.70264	275.51563	275.40576	275.34375	275.31201	275.3051P	275.30469	275.31226	275.33105
80° 04007 80° 04007 80° 04007				P0.00007	0000	0 0 0 0	80,95097	80,00007	0000	2000	ບິດດາດ	0.000.0	40,00,04	×>	120.68512	20220 2221	147, 43777	147, 79104	150.1349R	1024°14301	1004/5-40		150.30516	150.64711	159.70526	155.71854	150.67695	150.56052	CTAT DDCCC					0. P 7 R 7 N					0.46600		0.P6557		6555	0. HE5R6
8 5'8'IL 334.51025 734.13479	233.81226	233.56421	123.36841	71970.55r	232. 89273	02041.4462	332.712AD	122.67554	332.65625	332.65234	232.64710	332.65649	<u> 3</u> 32 . 667c7	7 SPAN	0.0	C.	19.19685	26.54990	43 DHI4F	44°744]4	12000-40 12000-40	71.77755	79-44824	86.66243	9012100	93.49217	00287.92	100.00000	=	150.70200	186.83635	215.94522	240.9460P	263.15332	302.13509	336.11654	366.68701	394.744]4	420.45074	447.386 ⁰ 6	47/•14966	468.61444	• 806F	46C. 747P0
007400 0.07400 0.05672				0.1=440	0.17309		0.70432	0.71783	0.23053	0.23461	0.74755	0.74P34	3.75470	S-VALINE	0.0	27910-0	D/ zz ⁰ °0	0.04413				0.12422	0.12083	0.15253	0.15861	3.16455	0.17024	0.17600	X-VALUE	-0-06000	00000-0-	-0.6000						-0.6330	-0.06300	-0.0000	-0-0000	-0-04000	-0.06000	-0.0000

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111/11	00000	1.00000	1.00000	00000	1,00000	1.00000	00000	1.00000	1.00000	00000	1.0000	1.00000	1.00000	1.00000	• • • • • •
T1/T1	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.0000	1.00000	1.00000	1.00000	1.00000	1.00000
(Da/na)	1.00000	1.0000	1.00000	1.00000	1.00000	1.00000	1.00000	1.0000	1.0000	1.00000	1.00000	1.0000	00000° i	1.00000	1.00000
Ud/ud	1.00/00	1.00000	1.00000	1.0000	1.00000	1.0000	00000 • 1	1.00000	00000 1	1.00000	1.00000	1.00000	1.0000	1.00700	00000
0P/(PT0-P)	-0.0447P	-0°02608	-0.01817	-0.01390	-0.01141	-0.00836	-0.00657	-0-00530	-0.00435	-0.00364	-0.01307	-0.002R2	-0.00258	-0.00233	-0.00206
rp/(pt-p)	-0.04675	-0.07417	-0.06742	-0.05615	-0.05308	-0.0494]	-0.04764	-0.04613	-0.04475	-0.04267	-0.04255	-0°04142	-0-04083	-0.0392A	-0.03687
57 A D	0.0	0.0	0.0	0.0	0.0	0.0	0-0	0-0	0-0	0-0	0-0	0.0	0-0	0-0	0.0
2 NN 05	1.40002	1 40002	1 40002	1.40002	1.40002	1.40007	1-40002	1 - 40007	1.40002	1 40007	1 40002	1.40002	1 40002	1 - 40002	1.40002

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STATION NUMBER 9 POWNSTREAM DE REMOTE

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٩٨	55.1027679			9125450	*1.******	77.6676782	20.0004785	15.7819462	11.5792279	R_1780500	5. 4433R70	3 2226273			LINTER C	-0.3189327		M V M	0.4412726	n.4509085	0.4400475	0.4675371	0.4737375	0.4820775	0.4885577	0.4913750	9.492342A	0.4019133	0.491 460	0.4908314	0.4907342	0.4909722	0.4917048		1.1140023	1.1004856	1 050606	1.1013861	FF15800.1	1.0036804	1.0008537	1.0R94175	1.0889740	1.0801478	1.089533R	1.0496950	1.0897446	1.0896735	1.0892487
×۲	147.29565	150 38770			1000 °C - 1000 °	10289 161	160,62175	162.39343	163.28696	163.56108	163.45753	3. 2146	163,11656		142 16031 142 16031	163.39151		33	0.46521	0.74655	0.81771	0.48114	0.93879	I.04140	1.13246	1.21572	1.70117	1.36227	1.47965	1.46220	1 . 40414	ທ ທີ	1.55654		7. A745 A	1.33144	1.17731	1.20472	1.32190	1.44650	3.19720	0.41457	-17.52277	-6.42792	-4.60045	-10.33R41	585.16309	6.59732	1151
>	147.29565	150.38770	153 21210				100.62175	E4595 -241	62.296	1 63.561 08	163.45753	163.21463		5	162,15071	5		X / M	0.4092	0.43104	0.44638	0.45764	0.44639	0 • 4 7 H H 4	D.4R675	0.40014	0.40163	0 • 49 j 64	0.49105	0.4907R	0.49072	0.4909	0.40170	EDC	21 06849	17.07266	14.00320	11.90725	10.103A6	7.50905	5.E7700	4.04645	7.86597	1.90830	1.1484	n. 90002	0.4880 6	0.1 APO7	-0.11184
RETA RAP	138.44377		145.74408		000054-14-1 09007 091					157.58072	158.93237	159.0701		140 87507	161 22627	161.58510		M<	0.44127	0.45091	0.46005	0.45/54	0.47374		•	0.49128	٠	10104.0	0.49115				0.49170	T APEA	•	5.52461	5.33778	122123	5.11714	10.07594	92438	9.84146	9.A0167	G. TRRG3	6110L°0	7.89085		4.80631	4.80266
	140.57603	4	146.56349	160 50300			156 5500	2000cr.+r.	156.202A0	157.61496	158.84308	159.91066	160.38785	160.87663	161-22638	16].58516			-166.15975	-198.44240 225 22055	[CDH/	-/4%./3430	-264.//100		-1961 -966-	-364,43852 201 10000	-396.42015	-422.12695	-446.17480	-457.1278	10100 014		-490°14780	TUT TEMP	7 R.R. 00000	288.00000	2 R.R. 00000	7 R.B. 00000	288 . 00000	2 R.9. 00000	789.00000	284.00000	288.0000	288.0000	288.00000	2 P.A. 00000	288.00000	a a	288+00000
A TUTAL	340.23535	340.23525	340. 73535	340 23535	340 73535				340.23535	340.23535	340.23535	340.23535	340.23535	3675, 23535	340.23535	340. 23535	:	3	222.04671		2023 COC		314/4°/16			403.87373	450°0°024	452.66919	475,09033	482°0°284			<pre><li< td=""><td>Trit poEcs</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>05510.1</td><td>1.01350</td><td>1.01350</td><td>1.01350</td><td>1 • 01 3 5 0</td><td>1.01350</td><td>1.01350</td></li<></pre>	Trit poEcs	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	1.01350	05510.1	1.01350	1.01350	1.01350	1 • 01 3 5 0	1.01350	1.01350
ALPHA	8°, 00097	P9. 00007	Pa. 99097	80 0001	80,00007	00 00001			/ honh "hk	89.000	89,09047	R9. C9CG7	89.95097	A9,00007	Fos, cgra7	89 . 00097			0.0									0.0	0.0					STAT TEMP	277 . 20386	276.74595	276.30371	275°c357a	275.62769	275.16211	274.87720	274.73242	214.68774	2/4.70459	274.74414	274.76050	274.75538	274, 75317	274.71533
ALPHA 940	80.0000	Rc. arcg7	80°0000	80,00CC7		0000	00 COC07			0000	89,99097	89, 6000 7	80,000 80	90, 90c97	R0.00047	8°,90°07	21	X	136.60048	100014041		155 334 43	150.24470	141 49474	167 07607	162 26721	14/00*00T	1010000	20/11/00	000040°01	26031.571	. 6		STAT DRESS	0.89465	0.88153	0.87661	0.87753	0.86912	0. 46400	0.86087	0.45929	U**C# •U	0.87858 0.05015	0.85941	0.85959		• 8595	0.85910
A STATIC	333 . 75736	333 . 52148	233.254AR	123.03320	737. A4607	327.56567	227.20265		122+ 304L7	332°27905	32.28921	332.31323	232.32300	332 . 32593	232.31860	232°29560		A STAR			22 421 20 22 421 20	21 02224	02007-E4	52 40443	62.60310	01c00012	11.00114 70 04030		10-0707 00 01010	72006 63	54-68C03				166.15875	198.44249	225.28951	248.73430	269.77100	307.01758	339.73975	304° 33802	570°0757	66921-224	440°1/4F0	451.12778		4 19 99951	490.74780
APIU	Ľ.	0.10272	3.11662	0-12976	0.13065	0.15802	0 17586			0.2002.0	0.71840	0.23003	0.23691	0.24274	J.24P44	0.25400	5 - V 1 1 1 5		0.01.22		0-04276	•	0.07292	0.08986	0.10517	0.11020	094210 094210			:-	17	:-		X-VALUE	-0*0000	-0.04000	-0*0*00	-0.4000	-0-0+000	-0*04000								000-00-0-	-0*0000

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	DP/(01250) -0.11250 -0.01455 -0.081440 -0.07465 -0.07465 -0.07465	- 0.05910 - 0.05910 - 0.05910 - 0.05915 - 0.05915 - 0.05915 - 0.0593 - 0.05	-0.04577									
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STATION NUMBED 10 NOWSTOEAN OF DEVITE

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VW 156.47755 157.83749 167.83749 160.13309 162.40349 167.3710	164,00015 169,54094 169,24804 164,24804 167,37501 166,75713 166,77733 166,77133 165,77133	1. 2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2.2	1.42757 1.46877 1.46877 1.57064 1.57064 1.57064 1.57064 2.5801 2.83861 2.83861	$\begin{array}{c} 1 & \mathbf{c} \\ 1 & $
v 154.47755 157.847755 157.843740 160.13300 167.40120 164.40340	168.00015 168.53004 160.368306 168.47732 164.75713 166.75713 166.75713 166.75713 166.47733 166.4673 166.4733 167.4733 167.4733 167.4733 167.4733 167.4733 167.4733 167.4733 167.4733 167.4733 167.4733 177.47335 177.47335 177.473557 177.475757 177.47575757 177.475757	2 2 2 2 2 2 2 2 2 2 2 2 2 2	0.50410 0.50737 0.50744 0.40021 0.40818 0.40818 1.46812 1.46812	14.02170 10.05170 10.05170 10.05170 10.05170 10.07552 1.05812 1.05812 0.1580 0.1580 0.1580 0.1580 0.1580 0.1502
RETA RA 130.25219 143.13076 145.74753 145.74753 147.311810 147.31157	1522 1552 1552 1552 1552 1552 1552 1552	0.4400 0.4400 0.47400 0.47400 0.48858 0.48858 0.49400 0.50420 0.51044 0.51024 0.51024 0.51023 0.51023	0.50414 0.50734 0.50734 0.600055 0.60072 0.60072 1.00 5.0019 5.019 5.00 5.00 5.00 5.00 5.00 5.00 5.00 5.0	R R R R R R R R R R R R R R
АБТА 141.35446 144.46144 146.65601 146.65601 146.15980 140.75190 152.04875	153,00284 153,50284 157,00284 158,51116 150,47712 160,45405 160,05185 161,05185 151,35087	WI -191.6156 -210.65125 -235.16512 -235.00627 -212.00627 -212.006407 -212.006407 -312.0163 -3446 -32471	-445.83496 -458.83496 -458.17334 -440.11521 -440.74780 -440.74780 -440.74780 -788.00000 -288.00000	788,0000 788,0000 788,0000 788,0000 788,0000 788,0000 788,0000 788,0000 788,0000 788,0000 788,0000
A TOTAL 340.23535 340.23535 240.23535 240.23535 240.23535 340.23535 340.23535	2460 2460 2460 2460 2464 2464 2464 2464		477.13677 497.57442 497.63228 507.62871 517.885500 1.01350 1.01350	1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350 1.01350
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AL PHA AA AC, TACTT AC, TACTT AC, TACTT AC, TACTT AC, ATCCT AC, ATCCT AC, ATCCT AC, ATCCT AC, ATCCT	70000 700000 7000000	145 VX 146 VX 150 22727 150 22727 161 40171 165 172317 165 172375 165 17275 165 17275 165 27575	167.31238 166.75096 166.77096 166.76865 166.24187 165.44187 7687171 0.87171 0.96886	$\begin{array}{c} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 $
4 Statif 332.0604 c 332.83278 232.43278 232.45280 232.45280 232.45280 231.45280	221.0 221.0 221.0 221.0 221.0 221.0 221.0 06118 222.0 1465 222.0 14655 222.0 14655 222.0 05005 223.0 05005	440400 440400 440400 440400 440400000000	85.75677 80.46300 93.05038 56.56117 56.56117 100.00000 100.00000 181.61542 181.61542	2355 257 257 257 257 216 217 212 214 242 343 513 447 253 513 447 113 447 456 251 57 3447 153 460 251 57 247 163 163 163 163 163 163 163 163 163 163
24071115 0.00490 0.10894 0.12972 0.12328 0.14334 0.14334	0.17787 0.17787 0.17787 0.216728 0.231011 0.23114 0.23714 0.24288 0.24288 0.24288 0.24289 0.24290	7-VA 0.0 0.0 0.0 0.0 0.0 0.0 0.0 0.	0.13727 0.14414 0.14414 0.14450 0.16000 0.16000 0.16000 0.16000 0.16000	- 0, 02000 - 0, 02000

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۳>	162.31140	145-10808	1 6 R - 66 3 R 5	171-47127	AOFF0 . 571	177.30542	178, 82000					140 2044A		166 67601	141.76057		35	0.76807	0. 82583	0.80935	0.05570	1-00-1	1.10030	1.1ª204	1. 25557	1.37299	[.38533	• 44 40	•4724	5001	•	•	P A D C	1.76261	0.05161	0.03016	1.08098	1.37296	2.87515	-22.89766	-2,19103	-1.10497	-0.71656	-0.5201 A	-0.45526	-0.40386	• 3619	
>	162.31140	165.10808	168.46385	771.47137	20 220 271	1 77 30563	178.83000	179.84811	177.52402	174.00455	171 37125	160.28666	166.00742	164.47691	141.76047			1 2 2 1	4 7 1 B	0.48845	0.50741	01414.0	1.50055 0	0°53701	0.53065	n.53627	1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1	10911.0	H2014°L	70705°0	0.48641		7 01	71 "36AQT	18,35083	1 5. A46 90	13.6RNGR	11.75ªA6	R.5370A	F, 01 R 1 l	3.75043	1 .05033	с. -	-0.56724	-0.07272	-1.780AG	- 47027	-1 5566.7
SETA RAD	140.52463	143,50395	145,58415	147.21400	149.57044		5	154.48502	156.07013	157.57327	159.0221	159.71991	140 40747	161.08551	161.75575			0.48830	0.49712				0.014 0.010	0.54079	0. 52,52					0.49513	. R . R	•	V Joëv	0.0	5.43222	5,78047	5.15498	5.04807	987 9.0	9.74307	9.70051	- 70806	2/1/1 D	9,49245		1//0000	5.10465	
RETA	147.511A3	144.92271	146.41195	147.96024	140.11730	151.12502	J52.ROR13	154.53362		157.57414	159.02324	159.72260	160.41200	161.09135	141.76202			x02/1°/51-	-224.JA406 -265 00000							-4/00.04.44	10.00°r	450 13135	-460-13010	-4R0.00744	-490.747RD		TUT TEMD	28ª , 0000	2 RP . 00000	289.0000	7 88,00000	7 HH 00000	2 8 8 00000	784,00000	788.00000	200 00000	200 00000	247.00000	χc	C C		ĸ
A TUTAL	340.23535	353	340.73535	340 . 23535	340.23535	340.23535	340.23535	340.23535	340.23535	340.23535	340.73535	240.23535	340.23535	3.5	340.23535	:			10100100	216 66065	233 66606	366 05233			-	F. F. OOF	78 60506				514.77314 .	•	TLT DRESS	1.01250	1.01350	1.01350	1.01350			000	1.013510						1.01250	
AH9.IA	70000 . PA	80°0001	R9. Groo7	R9.99997	80° 00007	Bo.cgray	89. COC 07	8°, 09097	R9. acca7	89,00007	89.0007	89. 00007	40,000,09	40°0CCC7	Ra.cgco7												0.0	0.0	0.0	0.0	0.0		STAT TEWP	274. P903P	274.4348]	273.97793		0x:47.47.2	64000000000000000000000000000000000000	215000 CLC	275.212072	272-76172	273.38623	973 12050	274-12500	274.53833	74.9.47	
ava vholv	80,00007	AC.00007	80,0000	R0.01907	80,00001	89.00001	Ro. rare7	80°0007	80°00007	40°00'04	80°00001	Ro. coga7	80,07007	ċ	89.30007	> 7	151.15202		۰.c	166-606-24	170.24277	175.24100	177. 09673	178-46510	177.42022	174,98737	171.36295	160.26710	166.04170	164.421 cR	161.70ccp		STAT PRESS	C. 96102	50078 50030						2 1 2 C 8 0					571	0. P6199	
A STATIC	332.40161	732.125cg	331.789A7	331.48047	33 1. 223A8	320.96507	230° 70053	330 . 69995	230 . 84253	731.11206	221.49097	731.7050A	RAAFP.JEF	27981.5EC	232.45435	N V D V L		8.89078	16-63319	23.55606	29. 85964	41.17615	51.26483	60.50121	6c.11945	77.27716	85.09012	88.89423	52.64180	96.34100	100000.001		=		767.10505 265 00540	0440°01440	284-71C73	317. C5RED	347.59276	374.72632	400-04346	424.007R1	446.95776	458.12125	469.1381R	480.00244	490.74780	
PANIUS													.24282		.25400	5-VAL 115	0-0	01251	0.02528	0.03541	0.04530	0.06759	0.07702	30 106	J.1 0506	0.11746	0.12034	0.13512	0.14082	0.14644	0.15200		X-VALUE									0.0000.0	0000000	0.0000		.00000	0.0	
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⊥(na/na)	1.00000	1.00000	1.00000	1.00000	1.0000	1.00000	1.00000	1.0000	1.00000	1.00000	1.00000	1.10000	1.00000	1.0000	1.00000
Pn/Pn	1,00000	0000001	1.00000	1.0000	1.00000	1.30000	1.00700	00000 • i	1.0000	1.00000	1.00000	1. CC000	1.00.000		1.00009
(d-gr0)/dr	-0. U7R36	-0°02894	-0.02809	-0.02451	-0.02455	-0-02053	-0.01680	-0.0123A	-0.01012	-0°0000-	-0.0036P	-0.00215	-0-00060	3 ,00096	0.00255
(u-14)/au	-0.07140	-0.04945	-0.10016	-0.10747	-0.11144	-0.113R5	-0.11150	-0.10507	-0.Fra30	-0.07425	-0.04590	-0°02845	-0.00Af1	0.01464	0.04143
ETAD	0.0	0•0	0.0	0.0	0.0	0.0	0.0	0.0	0° C	0.0	0.0	0.0	0.0	0"0	0•0
VMNVS	1.4 2007	1.40002	1.40002	1.40007	1.47002	1.40707	1.40002	1.47002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002	1.40002

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*>	211.75552	212. 42102	211.50810	200.67764	207.39000	204.8A510	204.24594	205, 20370	204.08152	208.9401 0	210.01251	210.60356	211.71306	213.04017	214.R7449		37	0.47578	20112.0	0°2760	0.74857	0.79070	0_84006	0. 80360	0.04773	0.00074	1.04574	1.07772	1.08322	1.00251	1.10322	.11.2		٥٩٦٢	-0.41850	-0.41108	-0.41419	-0.44599	-0.51638	-1.02587	11.04000	1.18344	0.71173	0.50273	0.35471	0.29780	0.24954	1.20731	9.1727R
>	BCCFB LLC	240.63270	747。85840	257.21143	252 . 5244	745.5290	241.14014	738.ANR50	727.07772	738.51300	740.97147	747.45814	744. RREAL	247 KOI NG	750 . 87 <u>6</u> 82		X/m	0.60475	r •40a22	0.4051A	0.50036	0°5000	0.59414	n_58100	0.58420	0.590.07	0.50347	0.50444	n 59476	0.50507	0.50703	0. K0085		202	A.17770	6.17144	5,02128	556734	5,12A13	4.17FP7	3,385,80	2.025QJ	7.7576	ן כבן א. כ	3,0723R	01700.5	CUEU3.5	4.01557	75572
ACTA AAR	114.34755	120.76085	124,09475	124.41602	131.25511	135.85573	130.24770	141.97859	143.9505	145.32204	146.29340	146.63480	146 00462	147.09018	147.18553		Ş	0.80073	. 7765	0.75613	0.73871	FR257.0	0.70187	0.68871	0.68087	0.67807	0.67874	0.69747	0.64613	0.40071	0.69659	r0201 0		8 AREA	0 •0	5.21514	10611°3	5037605	5.06565	10.12147	10.06799	CPARQ,	0.80757	C_ROSCA	0. 40245	4.97974	4.95002	4.97115	4.997.04
	115.47109	02700.001	125.13889	128.54774	121.36697	135,031A4	139.33720	141.01497	142,99158	145.35439	146.32144	146.67841	146.05648	147.15430	147.76860		ī	-100°38917	-126.43701	-148°36801	-166.29404	-1 81 . 99613	12801.11-	-237.37808	-261 50488	- 283. 32373	- 301 00654	TCCC8.215-	-310.95654	-274.82405	-279.71757	. *		777	338.JG0R5	Ουρος.λεε	335.1 9005	334,59095	334°49005	334 . 50085	334 59085	34 , 500A5	234.70090	225 49095	778 60005	340,69995	342 T9980	345, 30085	347. 59095
1 1 1 1 1	368.49194	367.57100	364.07651	366.45601	366.65603	346.40181	346.401R1	366.601R!	366.70006	367,19702	368.81494	JAG. PROKE	371.01416	177.74104	372.57179		3	08262 726			_					250 67540	347, 77008	378.44180	182 00700	387.72876	302 14674	206.41357			1.46137	1.66130	1.46120	1,66120	1.66130	1.66130	1.66130	1.66130	1.66130	1-66120	1-66130	1.66120	1.46120	1-46120	1.46130
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1.00000	11629.1	1.00000	1.16180
1.00000	1,63917	1.00000	1.16250
1.00000	1.63017	1.00000	1.16567
1.00000	1.63917	1.00000	1.1604
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(TA/TA)T	1.16562	1.16403	1.16424	1.16754	1.16719	1.16785	1.16250	1.16319	1.16567	4[07[.1	1.17604	1.17917	1.18729	1.1A547	1.18880
T1/T1	0.99200	0.99762	0E000-1	1.00119	1.00090	1.00090	1.00060	1.00119	1.00269	1.00387	1.00000	0.90677	0.99329	0,09078	0.99504
1 (ud/ŭa)	21029	1.63917	1.43917	1.63917	1.63017	11025 1	1.63017	1.63017	1 1029 1	1,63917	1053.1	7 [067.]7	1 1027.1	1.054.1	1 1027 1
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1 9-970 / 40	-0- 00001	-0.02537	-0.03517	-0.03806	-0.03740	-0.02824	-0.01619	-0.00480	0.00416	0.01127	10-10-0	0.02407	0.13027	0.73,809	19120.00
10-101/04	-0.00062	-0.02093	-0.03376	-0,04165	-0°04214	-0,04271	-0*05452	-0°01031	0.01031	0.07107	0.05532	0,07000	0.00010	0.11379	0°14295
FT AD	0.0	0-0	0.0	0*0	0-0	0*0	0•0	0*0	0.0	0.0	0.0	0•0	0.0	0.0	0*0
8 × × 85	1. Jopoz	1.35898	1.30001	50005°I	1.39903	1.30033	1.30004	1.30003	1.30001	1.30POF	1.30884	1.30877	02002°I	1.39PE2	1.30ps4

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V 241.07825	30027.055	740.25426	740.64445	241.41249	241.06277	737.75562	234 04634	720.420GF	227.08151	223 83553	222010225	220.60951	710,1878G	21°,00285	ХЛM	0.46780	0°44500	0,42160	0.61360	r.60073	0.57397	0.57475	۲ T Ņ	0.54445	0.55544	0,53757	0.52479	0.51554	0.50421	1.4077K	202	4.70KF2	1,4061	100000	7.54AD1	, 1 A A 7	1 . 6 20 A E	1.41758	1.46668	2 i i c 3 ' c	3.40AR7	4 ° 2 5 2 3 Ú	4.57411	4°87137	4,60217	5,12210
АЕТА ААР 134.30208	134.36046	134.3670F	134.56575	135.16238	127.19718	140.35617	143.00820	145.45479	147.48984	140.31546	150.2091	151.09048	1 51 ,05040	152.79001	>3	0.49674	0.69278	Ú*487°Ú	0.69434	0.68855	0.48757	757730	0 • 66545	0.45374	0.64211	い。63059	0.62463	0.61497	n.613a1	0.49931	₹ ÅDFA	0°0	4.49041	4.54874	4.67457	4.79165	0.8552	10.020.01	9.95694	0,96957	10.02412	10.10975	5.73990	5.30510	5.40304	5.51567
8574]34,30075	134.41765	134.49452	134.59405	125.9276	137.10845	140.36475	143°10004	145.48774	147.52874	140.39460	1 50. 7A7A6	15! .! 7632	152.74092	152,04212	5	-727.97044	-222,5541	-217.13447	-212 \$76499	-211 07456	-717.3052ª	-243.34082	-267.37402	-280.40015	-308.789A£	-277 . 50488	-228.31079	-223°00698	30	-363,02725	TIT TEMD	335,60005	335.50000	735,70080	335,09085	335.00000	Unpop . 455	7407040	335,0000	ろうちょんじゅつち	90000°285	338,6006	330,59095	340.50009	4	1000 2 . 42
A TOTAL 247.19071	367.08543	766,CROOK	346。87278	266. P] 934	346.76534	366.71069	366 PIC58	267,10971	4CF00-7AF	348.81404	249, 29517	340,77400	270,25293	370.79345	3	276.76243	317,88500	110. 574CA	202.40549	20150,800	206,22632	316.01460	234.2560	3674704	366.16805	10010.375	378.30615	381.4401 0	5135°7	386.77214	TUT POECS	0722701	1.43770	1.64470	1.44400	りやさらやい	1.66130	1.66130	0-149-1	1.66120	1.66130	1.66120	1.66133	1.66130	1.66120	1.66130
75.541 AC 75.541 AC	71.40750	-	63.52391	60 . 80766	56.61903	57,09061		50,40746	69-02380	5P.6P570	~	Ľ.	55.4447K	24.70940	0.7		76.20990	CI RONCO					120, 20000	115. 79060					<u> </u>		STAT TEWD	96.				306.0507R	306.03418	306.72070	3C7. 7002R	309.32544			315.09765		5.53	318 . 91248
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111	51.	50.94	50.	50.	0.5	50°9560	51.6671	52.4902	53.6474	55.0100	55.7180	56.4147	57.0AR1	57.7840	SPAN	c	4	1.5240	7.1720	2.765P	3.7986	1	4. 765C	63.01275	3.1133	80	833	502	5.51.90	0000	=	.8796	564	8450	0	6745	905	α.	0	^ :	-	0	6 . A	0	63.2067	71.4272
PANJHS 0.14000	1545	0.15055	. 3 6 6 3	107 i.	.181.	1101.	· 2004	- 20 c .	Jalc.	- 2270	2155.	1362	7355.	1740	S-VALUE	0-0	•							0.16372		•	•	•		•	1: IV-	٦,	7	٠.	-	7	-	۰.	٦.	٦,		-	0.15510	٦.		

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V/V 0.8704969 0.8795897	0.84065176 0.0133844 0.0314737	0.05%4143 0.0605677 0.07394390	0.9726101 0.9671502 0.9594991 0.9556733	0. 0573862 0. 0571867 0. 0543047
0/0 1.0445 1.09590	1 - 07 - 75 7 - 060 - 1 1 - 06 - 70	1.02074 1.02074 1.01776	1.01776 1.07018 1.02434	1.02762 1.02760 1.025582
[T1/T1]T 1.16562 1.16603	1.16424 1.16424 1.16319	1.16795 1.16750 1.16310	1.17014 1.17014 1.17604	
1,00000	1.00000 1.00000 1.00000	1.00000 1.00000 1.00000	1.00000 1.00000 1.00000	1.00007 1.00007 1.00007
(pr/pr)T 1.51]84 1.51589	1.62570 1.62694 1.63535	1.63917 1.63917 1.63917	1.63017 1.63017 1.63017	
01/00 0.08332 0.68570	C. 0001 0.00354 0.00585	1.00000 1.00000 1.00000	00000 00000 1.0000 1.	1, 50000 1, 50000 1, 50000
10/(070-0) 1.25378 0.10072	0.1578 0.11570 0.08640	0.04748 0.02914 0.02124	0.01ctR 0.02035 0.023227	0.02545 0.07545 0.07333 0.02333
17772 1.1773 1.17273	0.13010 0.13010 0.13010	0.7249 0.05349 0.04676	0.04850 0.05833 0.07180	0.074F5 0.79344 0.04458 0.09126
STAGE FEF 0.88623 2.02730	0.95893 0.95893 0.91732 0.92542	0.03513 0.93699 0.93699	0. 92055 0. 95789 0. 86598	0.85555 0.84240 0.82931 0.81527
500001 1.30502	50005 I	1.30003 1.30003	1.30004 1.30006].35977 1.30870 1.20862 1.30854

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3. 8376060		())+(),*)		11.80.0510	I 2. 75 4 550	26/9604-11	8666711.01	9.4075707	R. 0439602	9.4527912	10.2865166	11.3140602	52254	5.023516	19.1277924		5701		2 201200 00 00 00 00 00 00 00 00 00 00 00 00		0 577626	0 56 07 260	0.5571646	0.5419601	0.5237426	0.5779806	0.5105342	0.5014643	021004	47444	0.4454053	CH a	1.4158030	1.4231A34	.42RA19	1.4328403	1.4362783	1.4388046	1.4387312	1.4369860	1 4 3 5 6 1 2 7	1.6307552	1.4327824	1.4370526	1100001	1.470319	
211.61170	200 E00E2	500-5 -507 505			202.02644	20141/ 102	107.33850	1 ol. 70436	185.42766	185.60732	1 A1.7 7 7 1 9	178.9A076	175.37452	4725	1×7.00427	1								O DRUGE	0-07706	1.02271	1.04 400	1.05820	. 0454	.0779	1.07897		-0.38282	-0 • 54 76 R	-0.78658	-1.27843	-4.77895	1.05371	0.56975	0.53004	1. R0707	3,35,277		- 7.48370		513	
711 41170			x+114.40%	15-00-40z	2085.4024	700,05761	212.75040	216.11.02	71480.AJC	217. N3007	215.71520	214.01062	4120	211.02563	-04022°0UL	2012	2 505 60		. 1		0 57530		0 55663	0 56121		0.5220	0.50070	0.50046		•476g	0.44743	505	1.0400A	2.12113	2.86AD2	3,31140	2 ° C () 5 8 4	3.20878	12950 .	7. AI 001	7.76460	, a1079	17326 6	3.67634	rc 70r 7	5.37R34	
1 1 2 00000			146.45440	146.80870	146.30975	45.79453	145,85190	46.		140.24510	150.87405	151.71335	152.59670	153,49095	154.47751		7 1 1 1 1 1 1 1 1 1 1			0.07×0.0		17003 0	3454C*0		0.51500	0.61155	0.40441	0.50997	•	5993	0.59242	7 40F4	<u>،</u>	4.60608	4.63656	4.65989	4.64510	0.3A15A	0.55450	o (10.13476	10.3061 P	r C	5.30350	5,30694	5,1157 5,1157	,
8118 123 00223	t •	147.48609	146.49745	145.95733	146.44RH	145.83867	145,88602	144.47401	147.61568	140°27786	150.01383	151.76122	152.6521	153.58000] 54.57423		114 COC					75755°505-	-140°-04145		-707.07269	-711.01870	-276.25052	01204.555-	18261.828-	-343.78401	4[10].04c-	1 <u>11</u> 1540	235,60005	335.50000	ΰαοοι , ζεε	335,09985	335,0000	334. 89990	08001°765	335,00000	335,60005		338,69905	230,50085	340.50000	341.30000	
A TOTAL	1/121.05	167.08643	366.94096	366.07378	366.81934	346.76514	366.71069	366. מוָהקא	16001-775	367 ° 0308	368 81494	340, 7951 7	345.77490	370.25203	370.79345	:							100/x x x 001		345 02000 365 02000	367-56660	373.49608	277.47754	180.°0674	384.17554	347.11.770	TAT 99555	08013	1.61080	1.61980	1.67120	1.47670	1.63640	46! 0	1.55440	1.66130	1.46130	1.46130	1.66130	1.66130	1.66130	
AL PHA		Aq. nara7	40°0001	97 ° 76633	87.42151	74.75067	£2120*83	62.52862	58.20R07	58 . 74612	57, 504A7	54.64778	55.4CA11	E4. 22080	57, 77P74		0.0	0.0		0.0	A.00000	50000 FZ	00003 02	0000	114 70000		115, 20000	117.5000	120.000	1.23.00000	126,00099	CTAT TEMD	5.5026		314.10303	314.10156	313 . 84302	313.1º116	312 . 31641	-11. A0005	312.07520	313.69400	315.70044	216.95701	319.0590A	319, 29500	
ALPHA RAP	6 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	80°00001	Rc.conc7	1. TTO	R3.43370	74.77477	68.05736	62 55692	58.23700	58.77012	57.63718	56.70050	55 57361	54.34740				211.37695	20P 44 750	206.31P67	205.10560	204.6471	201.380c7		101.54374 105 21186	10E 34444	181.40022	179.5209	174.87000	170.73155	165 . 9c0 A2	CTAT DDECC	° C	1.28254	.2 A R	11205.	1,79411	FTFOC .I	1. 29005	1.2A536	1.29676	1 . 20/85	J. 20R10	1.30274	1.30700	1.31357	•
A STATIC	354. 94204	355.06689	355.191PG	355.19312	355.04712	354. 67ROK	354.18350	353,90014	354.04346	354.00283	354.07251	356.71582	357.38209	358.06616	58. R144		A SPAN	0.0	5.96611	11.75788	17.38425	20088.22	33.33667		53 . 33360	04°08031	1. 75446	86630682	90. 85325	95.41162	100.00000	Ξ	289.91177	300.88067	311.62451	322.06177	332.18262	351.65430	370.44165	388 . 74658	406.92373	424.41970	441.45972	449.90710	458.3330l	466.78491	
Antus	1 5030	.15573	.14120	.16569	£6121.	.18230	.1c173	20120	21056	21066	2284.8	10000	0.23722	24160	0.24600		S-VAL UF		• 30° - 7	0.01129).J1660	26120.0	002200	04172	0.05120	•	0.070.0		.08722			- V AL 116		17530	0.175.00	1.17500	.17500	.17500	0.17530	1.7530	1.17500	.17500	.17500	0.17500	.1.7500	0.17500	•

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v/v 0.4760427 0.8700699 0.8598281 0.8543945 0.8549863	0.8672123 0.8948290 0.9233808 0.9464360 0.9464360	0. 9614895 0. 9637567 0. 9627833 0. 9697833	, -		
0/0 1.06918 1.06968 1.07166 1.07341	1.06817 1.05559 1.045559 1.045559 1.04552 1.04552 1.025152	1.02116 1.01995 1.01920 1.01938 1.02938			
(TT/TT) 1.16567 1.16403 1.16474 1.16356 1.16350		1.1766 1.17617 1.18770 1.18847 1.18887		•	
T1/T7 1.00000 1.00000 1.00000 1.000000	1.00000 1.00000 1.00000 1.00000 1.00000	1.00000 1.00000 1.00000 1.000000 1.000000			
(PC/PC+T 1.59822 1.59822 1.59822 1.59961 1.50561	1.65460 1.62417 1.63236 1.63236 1.63217	1.63917 1.63917 1.63917 1.63917 1.63917			
pr/pr 0.00155 0.08407 0.08436 0.08436 0.08436	0.0000 0.00595 0.00595 1.00000 1.00000	1.00000 1.70000 1.70000 1.70000 1.70000			
ГР/(РТР-Р) 0.09271 0.09045 0.10798 0.11768 0.12154	0.1145 0.0145 0.01454 0.014345 0.01454 0.014540	0.02068 0.01907 0.01816 0.01806 0.01806			
10/107-7) 0.19665 0.10044 0.19462 0.19462	0.1542 0.15442 0.12168 0.09685 0.07880	0.06838 0.06604 0.06517 0.0517 0.05157 0.07157			
51 А5F Еге 0.0 0.0 0.0 0.0					
ТАМЧА 1.20000 1.20000 1.30000 1.30000 1.20000 20000	1.3000 1.30000 1.30000 1.30000 1.30000 1.30000 1.30000000000	1.30894 1.30877 1.39970 1.39952 1.39952			

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4>	-0.9665785	1.0307685	2.8031292	4.4876788	K. 2425556	0.7760028	12.3427410	12.5908575	17.0710830	1 2, 1 207592	11.3482323	10.4810190	0.3041570	P. 7637049	7. 2030640	2 V 12	n. 5536724	η. κες 3030	0.5494654	0.5457312	0. 541 4085	0.5342854	0.5285410	0.5275342	0. 5199736	0. 50034R0	0.4005247	0.4715538	0.4436560	0.4775F72	0.4027J69		1 4483727	1 4501524	1 4537780	1.4569054	4403586	1.4663134	1.4709969	1.4745984	1 4748343	1 . 471 7751	1 4477206	1 AAAAAAA	4666319		1.4720137
¥>	107.35A7A	96.8777	105.82187	104.51960	103.13246	100,60461	189.63507	1 88.29600	185.62745	187.10957	175.72115	169.12285	150.35894	152,34392	2858	ļ	7,09366	00000	1 03460	1.0594F	1.08151	1.12675	1.1754	1,16210	1.13841	1.12541	01201.1	1.08537	1.05007	1.0550.1	1.06756	RADE	-28-10246	-1-54166	-0.00387	-0. 94470	-0. A0693	-0.4380	-1.30809	-2.52741	5_3597R	2.02254	7.83333	-16 14497	46570 0-	15005 0-	-0- 22353
>	107.35878	196.9773	105,83187	JAK	102.12246	1 900, 6046 1	1 88 6 57507	180.51052	191.60361	1041.1421	106.43512	107.14017	104 60401	105,73505	TORAT . TO L	XNF	0.55347	0.55730	14047.0	0°54550	n.54147	0.52358	0.52741	1.57616	0.51855	0.50827	0.48050	0.47045	0.44700	0.47772	n.40222	202	-0.28061	0.20746	0.82014	1.32049	1.45277	4cù4u°c	3.75143	4.13675	12700.4	7,91044	2,70770	3,55306	3.24855	1,1050	2.44141
857A 848	145.74552	46.94226	147.02412	148.06701	140 0471	151.49385	153.18230	152,02094	152.83064	152.10037	153.70016	154.24877	155.23418	156.34144	157.68359	24	0.55367	0.55730	0.54047	0.54573	0.54170	0.53479	0.57854	JOUL O	0.53657	0.54711	0.54835	0.54967	0.54760	0.54255	0.53142	AOFA	0-0	4.59313	4.48048	4.70776	4 . 7 29 54	0 . 52034	0.61226	9.64176	0.70794	0,97200	10.14391	5.26568	5,52012	5,87510	<pre>< 07705</pre>
95 T A	145.74500		147.92680	148.06964	140,05541	151.72537	152,23184	153.0076	152,08754	153.15173	153.74760	154.79182	1 55 , 771 44	156.37263	157.70934	13	-280.81177	20075.105-	-312.47534	-323 , 25052	-333 . 7404R	-353,90030	-373.15112	-270.01172	-361.45796	-358.96362	-355,54590	-350.60A15	-245 . 42578	47	-353,95532	TIT TEND	335.69005	335.50000	735.70090	335,0908 5	335.00000	34. 99090	34 . 79980	335,0000	315 ,49095	00000	334 69C05	330,50085	340.50000	10005 175	347.30000
א דחדמן	15091.785	247.09643	366 69096	346 P7379	366.81034	366.76514	366.71060	346 PIOFR	367.10071	367.07308	368. 81404	360,20517	360.77400	370.25293	370 . 78345	3	350.67064					401.56460	419.12109	415.16323	406.51221	402.51543	304 . 5CRR9	389.74660	380.4130C	44	02214-285	701 00 FCF	1.619RD	1.Florn	1 FIGRO	1.41980	1.640	1.41089	1.410RO	1.67570	1.67564	1.4340	1.45160	1.45590	1.45950	1 . 4 4 1 2 0	1.55130
AL PHA	80 , 09097	40° C C C D D	7 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	89. anca7	5000	89. cor97	40° 00001	83 . 45893	75. 41247	64 85 7 8 4 8 5 7 8 4 8 5 7 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	63.40277	50°03085	54° 36682	51,024747	00112.04	(I A -	0.0	0°0	۰ 0	0.0	0• J	0.1		21.50000	47.5000	67.30009	P7.7000	101.20995		0000°6	125.20000	SIAT TEMD	355715	316.24002	316 , 750cR	3].6.20405	214.47107	316.95357	317.12427	217.15040	317.454EO	319.27246	219 . 53931	320. 20371	12502-125	7.470	374.14478
ALPHA RAD	80,00007	8°, COCQ 7	6.100	Ro. greg7	8c° aqaq 7	80° COOC 7	80,07097	82 44572	75。64610	£1009°09	63.45070	5c.0794R	12211 * 75	⊑1. 29864	40°24407	×	197.3559]	ŝ	lcc.pjj7c	104.46703	1c3.03154	190.35258	188.23174	197.79657	JR5.16864	181.70500	175.35432	æ	5	2.	1771	STAT PRESS	1.31545	3915	104	~	• 3 2 6 8	1.333An	•	•	2 6 4 7 2 8	•	1.34449	3485	1. 35287	5	27075
	356.45313	71905.30917	356.40430	356.43677	356.53125	356.74683	356. Aoc41	356.01895	357.08740	357.53467	358.230°6	358.65015	350.19458	350 . 84717	360.76416	T SPAN	0.0	6.09172	11.96654	17.66103	23 . 19554	37 . 84218	44.009an	53.70523	. 63.02431	72.11116	R1.0R076	A5.60327	90.26210	55°07573	1 00• 00000	=	289.81177	301.34937	312.47534	323.25962	333.7404R	353.9079	373.151.2	351.51125	409.15706	426.36377	443 . 345cF	451.9C820	460.72583	ŝ	79.1
2117 ng c	0.1 5000	7.555 2.6	0.1517P	0.16721	57273.C	0.18717	0.jcjla	5,2262	JTILC.C	0.22967	9*25ctV	0.23389	9	2,24217	0.24800	S-VAI UF	c•c	0.00507	0.01173	1-710-0	c122C.0	715FC.C	0.04313	J. 05763	0.06176	0.97067	0.07546	0.0838C	9 d d d () •	5	0° veb0v	-VAL UF	.1 0500	0 € Û Û	.10500	.1°=0 <i>n</i>	•1°F0∩	.19500	.19500	.] 0500	•1 of 00	.10500	.19500	.19500	.1050	. 050	1 05

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0/0	BCCED.I	1.02641	1_02418	F 0 F C 0 - 1	1.02531	1.03104	1.07817	1.94341	1.04506	1.04183	1 - 7 - 0 - 1	1.03523	1, 73438	1.03543	1.03876
(11/11)	1.16562	6403	1.16424	1.16354	1.16210	1.16795	1.16250	1.16210	1.16562	1.17014	1.17604	1.17017	1.18220	1.18547	1. IRARO
T1/T1	1 - 00000	1.00000	1.00000	1.0000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.00000	1.0000	1.00000	1.0000	1.00000
1 (na/na)	1.50877	59872	1.50922	1.50822	1.59822	1. 50822	1.59822	1.40502	1.61322	1.6151	1°42°60	1.65375	1.63641	1 43017	1.63917
Ja/ Ja	1 - 00000		1.0000	91000.0	0.00576	0, 980 A6	ŋ.º4402	1. C9376	0.99417	0,08025	0.90415		ไร้ชยว"0	1.00000	1.0000
(d-010)/du	0.73521	0.02763	0.02384	0.02337	0.02551	0°037R0	0.04357	0+75154	0.05347	0°04434	0.03702	0,0347A	0.032¤6	0.03320	0.03592
10-101/04	00110	0.10120	P.C933A	0.00395	0.09449	0.11710	0.13420	0.15311	0.15455	0.14577	0.13303	0.12R00	0.12795	285°1.0	9.1496P
STAGE FEE	0.0	0•0	0.0	0.0	0.0	0.0	C • O	0.0	0.0	0.0	0.0	0-0	0.0	0.0	0.0
ビットロン	1.3093	1.20808	10000-1	1.39c03	1.30003	20002-1	1.30004	£0032°;	1°30cVI	1.39006	7 appa [1.30877	1.29970	1.3cp62	1.39954

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	0.1500 0.1517 0.15717 0.15711 0.15777 0.17335 0.17335 0.17335 0.2777 0.2777 0.27375 0.27355 0.27355 0.27355 0.27355 0.275555 0.27555 0.27555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.275555 0.2755555 0.27555555 0.27555555555555555555555555555555555555	357.52734 357.40088 357.28149 257.17454	80°00007	99, 66597 89, 66597	367.19921 367.08662	147.11745 148.13700	147.11740	1.97.36256	187,34756 187,54880	n. 2695A13	
	1 6417 16211 16781 16782 17336 17336 17336 17336 70272 70272 70272	357。40098 357。2914 9 257,17654	Ro. acra7	89 . ccr97	247,08643	148.13790			187 54 890		
	16211 16782 17336 17336 17336 19306 70272 70272 70272	357.29140 257.17454					148.13695	1 A7.54PAD		1.6701412	
00000000000000000000000000000000000000	16782 17336 17336 17336 17336 20040 20127 20172 20157 20157 20157	357.1765	Rc. gaac 7	40°0004	366.08096	1 40.07646	140.07330	187.64789	187.64789	2.0161844	
	17336 18306 10406 70217 21277 27215		80,000c7	Bo. accc7	346.87378	149.95259	140.94717	187.61234	187.61234	3.0193083	
	18306 10404 20272 20272 20272 20272 20275	357.14038	σ	89.00007	366.R1034	150.78073	150.77319	187.40472	187.40672	4.6496964	
	10404 20272 21277 22215	357.21924	99 , 0007	Ra. ccaa7	366.74514	152.37178	152.36145	1,84.11627	186.11627	<. <156155	
	20212	257.40723	R0.0000	80° c dc c 1	366.71969	152.00163	153, 89925	1 82.75207	183.75307	6.0790310	
	21252	357 . 8090A	89,00007	Aq. aona7	366. Plo59	155.33406	155.31696	190,00147	1,80,90147	7.1678419	
	22215	259 ° 47046	AC. 900C7	89.09C97	367.10971	154.42045	156.59557	17P.10008	178.1°078	4000621 a	
	10000	359.42358	80,09097	90°09597	367 , 00308	157,75441	157.72353	175.83439	175.83438	9.7396412	
	23045	360.38721	85 . Gogla	R5.09097	369.81404	157,05242	157.92201	175.54716	174.90514	9.5120106	
	ろうちく	360.R405P	R1.07417	91.06378	349.20517	157,88493	157.96122	175.05252	173.92275	R.4445006	
	73057	361.29076	76.96732	76.95061	349.77490	157,88031	157.87714	176.40703	171.94592	6.0134153	
	24375	61.71 92	• 5703	72 577AS	370.25703	157 . 00646	157,99472	77,02077	168,00041	2.2256823	
,	74800	362.25513	69.30307	61105.63	370.78345	159.21082	15P.30A75	177.12463	145.50362	0	
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000000	-VAL HE	T SPAN	××	ΩΛ	3		>₹	XVP	75	¥ < ¥	
	,	0.0	187.36737	0.0	345.10205	-280.R1177	0.52405	0.52405	0.06525	0.5240510	
	0.0417	6.300PR	36143 64136	٠	355.78003	00647.005-	0.52474	0.52474	10700-0	0.5747575	
	0.01211	I 2. 3542P	187.62 ^E 71	c•0	365.11694	-313, 20703	0.57521	0.52515	FPICO.I	0.5252102	
	E9710.	18.19241	187.57130	0.0	374.62646	P1545.495-	0.52527	0.52515	1.04RA6	0.5257679	
00	A5520.	23.83966	187.34002	0.1	283,010,582	-334.95728	0.57474	Q.5245A	1.07470	0.5247424	
•	90260.	34.64812	1P6.03453	0•0	401.205A1	-355 . 42505	0.52101	0.57079	1.1214	0.570141	
	•04404	44.93684	1A3.6524A	0.0	417.51831	-374.90945	0.51412	0.51285	1.lAR19	0.5141280	
•	.05372	54°81644	180.75940	0°0	473.10556	-393.6154ª	0.50558	0.50°1A	1.21060	0.5055910	1
• c	06307	64.36C55	177.07430	0•0	548 CO473	-411.68678	0.49709	0,49640	1.25141	0.4970944	
• C	•07215	73.62743	175.56442	0•0	462 B4085	-420,23971	0.4897	4 P B	1.29054	0.4807121	
0	င္ရွ္မ	82 . 6C875	174.64629	15.00000	465.35425	-431.22413	0.48711	U-48461	1. 29126	0.4853256	
•••	.08526	86. acpr2	173.61751	27.29000	461.25073	-427。24463	0.48762	0.48115	1.27977	715	
	< 36 d0 *	1.3420	171.84073	0	456.59154	-422 04720	44.		1.26379	0.4759343	
	- Úc 3 2 5	c (αu	3.0000	450.79712	r L	70b7°	446	462	0.466063R	•
•	00×60.	1000000001	165.67612	62 * 5 cddu	448°20080	-4]6.55542	0.48905	0.45735	1.23752	9.4573948	
×	-VALUE	=	5717 D0F55	STAT TEWD	TUT DRECC	TOT TEMP	A D C A	503			
•0	.21500	289° A1177	1.34351		1.61080	325,69005	0-0	0.08244	4- 03020	1 4702300	
12.0	.21500	301.74390	1.24284	319.02c30	1.41980	235.50000	4.84717		4.00601	1.6707002	
C	21500	213.20703	1,34241	217.809A1	1.61080	335.200RD	4.940az	0. 80045	A. 28051	1 4712467	•
•	.21 =00	324.26319	9536	317.61597	1.61980	335, NCOR5	080FA.4	1,10705	- 27 . 97955	1.4720840	
•	71 FOD	34°05728		317.55420	1.61080	325,00000	4.R4103	1.42170		1-4720071	
	.21500	355.4 2505	14632	317.69336	1.61080	334.80000	9.70365	1.69873	-1,11500	1.4760723	
ċ	.21500	374°00845		31 A. 02759	1.61080	CROPT . JACR	0,77153	1 . 805.84	-0-69820	1.4415187	
• •	.21500	393 . 61548	.3605R	318 . 7443R	1.61089	335,00000	0.47350	2 . 7 T R 2	-0.5076	•	
ċ	.21500	411 . 59628	€:	710.92847	1.41980	335 , 40005	0,00477	2.80673	-0.61022	•	
C	.21500	429.23071	1.77543	321.54429	1.61CRD	337.0000	10.13369	3.17530	-0*4540-	1 4894676	
•	.21500	446.23413	lear.	23	1.62530	338,69095	10.27409	2.11750	-0.61633	1.4PP495	
C	.21500	454 * 544FP	1.38505	324.22827	1.62940	330.50085	514215	2.78452	-0.64604	4470755	
0	-21500	462.76733	1.30782	325.03638	1.63360	340.50037	5.1814O	2.00420	-0.71100	1.4871950	
	Ľ.	К. 4	1.39050	25.	1.43770	0000£°14E	5.25300	0.754aa	-0.80774	1.4843768	
•	21500	479.15552	1. 30740	326 . 87 950	1.64060	342.59090	5,35222	-0 . 43741	-9.02074	1.4449706	

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d/ a	1.01988	1.01739	1.01461	1.01204	1.00932	1.01002	1-01330	1 01 70 2			00200	001001	5 u C 7 0 T	1.02235	1.01654
1 (17/17) 1 12222	1.16403	1.16424	1.16354	1.16310	1.16285	1.16250	1.16319	1.14562	1 1 7 0 1 4	1 7604	1 1 7 0 1 7			1 • 1 5 1 4 7	1.1 8880
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(01/01) 1 . 59822	1.50822	1.59922	1.59822	1.59872	1.59822	1.59R27	1.59822	1.50822	1.59822	1-60365	1.60770	1-61184	1 41500		1.61.875
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CP/(DTP-D) 0.02490	0.02174	0.01790	0.01420	0.110.0	0.00173	0.00750	0.01015	0.01446	0.01889	0.02289	0.32464	0.02509	0-02192		10410.00
10-10220 0•09220	0.08434	0.07540				0.04745	0,06790	0.08287	0.10245	0.11444	0.11356	0.11435	0.10002		
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a۸	-0.0647919	0.4271557	0.0159402	1.4021158	1. 8947868	7. R065958	2.4068441	2.4361267	2. 8837843	1.9510231	1-0704622	0.7745374	0. 5857677	0.6071716	0.4373430			Glovels o	0 51 231 66			0 EV3/200	0.5053860		0. 5002235	0.4966415	0.4078895	0.4011544	0.4808370	0.4802142	.4004Å	C10	1.4777880	1.4700812	1.4806175	15657941	1.4836003	1.4858904	1.4879907	1.4AR5507	1.4874930	1.442740	1.4704389	1 4747208	CTCTFT4.1	1 4701096	1.4654570
~ ~	183. 87982	1P2.62329	183.27748	182.82623	182.36770	181.50876	1P0. 74690	180.05258	179.26534	178.38026	177-53531	177.17015	176.94550	176,99503	177.47614	ļ			1.01525			1 1 1 2 2 4	1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 -	1.71784	1 75674	1_70648	1.33540	•	4022E 1	1.30143	1.41065	PADC	-10.34649	-4.67947	11497	-2.52531	-2.37RK4	11022	-2.67410	-2,14582	-1.45036	-1.17021	-1.16488	-1.20184	-1.95147	-28.14140	1 58325
>	1 R2. 97987	555,632	1P2.2774R	α	142.26720	1 P1 SCAPE	1 RO. 74690	1 80.05759	170,74534	378 . 3A026	177.53531	177.17015	176.04550	176.00503	177.67614	2011	2 C C C 2 T C		0.51231	0 51116		0.50738	0.50500		0.50017	1.40661	0.407BR	9.40115	0.48083	150420	1.4809A	E D C	-n_02019	0, 1 32 7A	0.7¤£3E	12052.0	0.50531	ר אפריק	(UUaU-1	1.00250	0,00174	N. K7K5R	0.34547	n. 75112	0.22206	0.0754g	0, 20415
RETA BAP	147.60567	148.69096	149.6407	150.62505	151.48521	153.0157A	154,33685	155.40312	156.53426	157.48332	158.34163	158.73642	159,09014	150.41704	150.67566		0 51270	0.010	0.51231	0.51115	0.50040	0.50744	0.50524	0.50309	0.50023	0.40654	0.40290	0.49115	C9084.0	0.49031	0.48997	A AREA		4,01960	4.02050	2,00643	4°2204	0.88740	0,01735	0.94616	0,09461	19.04087	10.11002	F.0A351	640065	26211.3	5.11A65
A T T A	147.60567	α	140,49435	150.62574	151.48653	153.71857		155.40776	156.53600	157.48453] 50,00070	150.41721	150,67577	Line .	-280 01177		-212.55850	-274.70663	-335.67261	-256.47330	-276.18250	79530,205-	-412 15801	-430,2045R	-447.07202	-455,27344	-462.35449	-471.21641	-470,]5552	471 TA40	335,49095	335.50000	115,7009	335, Jaoge	235,00000	344 90000	14001°455	335,00000	335,60005	00000-122	328 ,69095	330,500R 5	340.50000	141,10000	U0002°272
א דהדמן	10001.745	267.08642	366.09096	366.R737R	366 . R1024	344.75514	346.71040	366 81058	347.19971	367.00309	368° R1404	360.29517	369.77490	270,25203	370 . 78345	2		353.27622							450.18921		4A1.03247	498.53149	405,90707	513.45450	sln.oktea	TOT DRECS	UAPI',	1.41ºRD	1.61080	(AC14.1	1. K1 9 RO	. 61	1.619RD	1.61089	1.610RJ	1.41980	Udoiy"i	1.41980	1.61080	1°61ca)	1.61980
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АГРНА РАР	80°000	Ro. cocc 7	4c°c0rc7	ga, cara 7	0.00	80,00007	0000	P0.001C7	0000	Rc.orrc7	0000	0000	500	ga crca	ko, cara 7	~7	83.870B	183.43787	P2.2701	1 27. 87085	182.25736	1 81 48656	180.71478	180.01979	170.74214	3636	177.52290	177.16844	176.94426	176.00455	177.47501	STAT PRESS	1.25304	1.35356	1.35430	1°35275	1.35462	1.250P7 .	1.360A5	1.362P5	1 • 3 C 2 C C	1.36873	1.7714	5727F.[1.37407	1.27520	1.27482
A STATIC	357.98764	357. ROE15	357.73438	357.66543	357.66040	357 . 60180	357.712RO	257. 99453	35236,36228	359.17310	760.192P7	360.721c2	361.23657	~	362.2070	· CDAN		6.30100) r .	8-4742		35.20268	45.61170	55.53004	65.03346	74.19006	R3.051R3	P7.38400	91.652P5	95 . A5AP1	1 00•00000	=	289.83177	301.91650	3]3.55,840	324.79663	335.47261	356.47330	276.1 R3E9	304.06287	412.95801	430° 29468	47.0	734	63.3544	5.15	LC .
PAPTUS	•		0.531.0	0.16810		-	<u>.</u>	3.20442	5	17525.0	0 E i E Z * C	ີ	350	9 24 2 .	0. 24P00	5 -V N 115	i i i	0.00626		0.01410	0-02373	3.02450	ိုင်	0.05447	· C	I1270.C	-	ç	.0A9F	• 0 co	0 * 006 00	×-∧ אך ווב	•	0.74000	•	0.24000	0.24000	0.24000	0.24070	0.24330	0.02.40.00		£00	000	0.24100	• 2 40 0	0.24300

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APPENDIX B

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GEOMETRIC CONSIDERATIONS FOR SUBSONIC LEADING EDGES ON TRANSONIC ROTOR BLADES

APPENDIX B: GEOMETRIC CONSIDERATIONS FOR SUBSONIC LEADING EDGES ON TRANSONIC ROTOR BLADES

We first note that a simple leading edge configuration is obtained by sweeping each leading edge element dl in the plane formed by the local relative velocity W_1 and the radius (W_1 -r plane). This plane intersects the Mach cone along two generatrices that form the Mach cone angle μ with W. Any other plane through the apex cuts the cone along generatrices forming a smaller angle μ " with W in the W-r projection. Since the radial projection of dl is essentially proportional to sin μ ", it follows that the simple case defined above yields the shortest possible swept blade length for a given annulus height and a given relative velocity distribution W(r). For structural reasons, however, the leading edge must be swept aside from the W_-r plane.

The general situation is shown in Fig. B.1 (Refer also to Fig. 17).

la shows the projection on a plane perpendicular to the radius passing through leading edge point P. The velocity triangle is projected in that plane for visualization convenience.

lb shows the projection on the W-r plane, with the Mach lines forming the Mach angle μ with W. In general, W forms an angle ε_{W_1} with plane la.

lc shows the projection on a plane perpendicular to W, intersecting the Mach cone along circle c.

ld shows the projection on a meridional plane.

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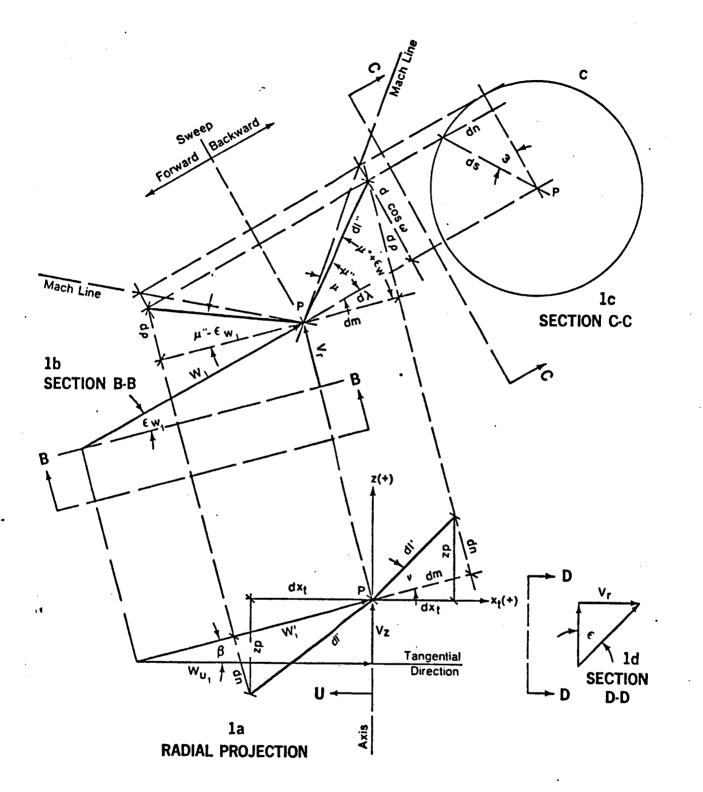


FIG B-1 SONIC SWEPT LEADING EDGE ELEMENT

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On la, the leading edge element dl has the projection dl' and the radial planes passing through dl and W form the angle v, which is a design parameter to be selected so as to minimize the blade bending stresses. The resulting lateral sweep component dn appears also on projection lc and causes the Mach angle μ between dl and W to project into the W-r plane lb with a smaller aperture μ ".

From la $dx_t = \pm dl' \cos (\beta + \nu)$ $dz = \pm dl' \sin (\beta + \nu)$

In the above and the following relations, the top signs denote backwards, the bottom signs forward sweep.

Since

$$dl' = \frac{dm}{\cos \nu} = \frac{d\rho}{\tan(\mu'' \pm \varepsilon_{W_1}) \cdot \cos \nu}$$

$$\left[\frac{dx_t}{dz}\right] = \frac{\begin{bmatrix}\cos\\\sin\\\sin\\\mu'' \pm \varepsilon_{W_1}\\\psi_{H_1}\end{bmatrix}}{\tan(\mu'' \pm \varepsilon_{W_1})} \frac{1}{\cos \nu} d\rho \qquad (B.1)$$

From 1b and 1c

$$\tan \mu'' = \frac{ds \cos \omega}{d\lambda} = \tan \mu \cdot \cos \omega \qquad (B.2)$$

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and since

 $\sin \omega = \frac{dn}{ds} = \frac{dn}{dm} \frac{dm}{dl''} \frac{dl''}{ds} \frac{\cos \omega}{\cos \omega} =$ $= \tan \nu \cdot \cos \left(\mu'' \pm \varepsilon_{W}\right) \frac{\cos \omega}{\sin \mu''}$

Therefore,

$$\cos\omega = \frac{1}{\sqrt{1 + \frac{\tan^2 \nu}{\sin^2 \mu''} - \cos^2 (\mu'' \pm \epsilon_w)}}$$
(B.3)

which is introduced in Eqn. B.2, yielding

$$\tan^{2}\mu'' = \frac{\tan^{2}\mu}{1 + \frac{\tan^{2}\nu}{\sin^{2}\mu''}} \cos^{2}(\mu'' \pm \varepsilon_{w})$$
(B.4)

Developing cos (μ " $\pm\epsilon_w$), Eqn. B4 is reduced to a quadratic equation for $\tan\mu$ ". The solution is

$$\tan \mu'' = \frac{\pm \sin \epsilon_w \cos \epsilon_w \tan^2 \nu + \tan^2 \mu (1 + \sin^2 \epsilon_w \tan^2 \nu) - \cos^2 \epsilon_w \tan^2 \nu}{1 + \sin^2 \epsilon_w \tan^2 \nu}$$
(B.5)

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(Only the (+) sign is valid in front of the radical, since tan $\mu^{\prime\prime}$ must tend toward tan μ when ν tends toward 0).

We define the blade profiles and their stacking in cylindrical coordinates. The angular abcissa of leading edge point P(r) then is

$$\theta_{L}(\mathbf{r}) = \theta_{L_{1}} + \int \frac{dx_{t}}{\rho} = \theta_{L_{1}} \pm \int \frac{cos (\beta + \nu)}{tan(\mu'' \pm \varepsilon_{W_{1}})} \frac{1}{cos \nu} \frac{d\rho}{\rho} (B.6)$$

$$f_{M=1} = r_{M=1}$$

$$Z_{L}(\mathbf{r}) = Z'_{L_{1}} \underbrace{\int_{1}^{t} \frac{\sin (\beta + \nu)}{\tan (\mu'' \pm \varepsilon_{W_{1}}) \cos \nu}}_{m = 1} d\rho$$
(B.7)

where

 $\varepsilon_{\rm w} = \arcsin \frac{\rm V_r}{\rm W}$

(B.8)

Eqs. (B.6) and (B.7), together with (B.5) and (B.6), determine the cylindrical coordinates of the profile leading edges, for a blade with sonic leading edge.

With the section profile data, the stacking of the centers of gravity is determined, and the blade bending stresses can be calculated. However, it is advisable to iterate the leading edge coordinates until a favorable alignment of the CG's is achieved, prior to the calculation of stresses.

B-5

By optimum selection of the lateral sweep and of the radial location of the point of sweep reversal, it is expected that the additional stresses affecting the subsonic leading edge blade will be reduced to:

> (a) Additional centrifugal stresses from the added blade mass necessary to materialize the subsonic leading edge configuration.

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(b) Bending stresses from moments without substantial component in the direction of the axes of minimum inertia.

APPENDIX C

FUNDAMENTAL ACOUSTICAL ASPECTS OF STATOR DESIGN

APPENDIX C

FUNDAMENTAL ACOUSTICAL ASPECTS OF STATOR DESIGN

C.1 Continuous and Discrete Line Sources in a Stationary Acoustic Medium

Continuous Line Source

Consider a line monopole source of the type

$$i(k_{o}x-\omega_{o}t)$$

$$q(x,t) = Q_{o}e , \qquad (C.1)$$

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where Q_0 is the source strength per unit length. The line source of Eq. C.l represents one wave traveling along the x-axis (see Fig. C.l) with a velocity c_0 given by

$$c_{0} = \omega_{0}/k_{0} \qquad (C.2)$$

One is interested (only) in the far field pressure p(x,y,z,t) radiated by the line source. Define

 $r = (y^2 + z^2)^{\frac{1}{2}}$ (C.3)

Consider the case (referred to as Case No. 1) where the source velocity c_0 is supersonic, i.e.,

 $|c_0| > c$, (C.4)

or equivalently,

 $|k_0| < k_{a_0} = |\omega_0|/c$. (C.5)

Here c is the sound speed for the medium and k_a is the acoustic wavenumber at frequency ω_0 . For this case, the⁰far field pressure p(x,r,t) is non-zero; in other words the line source can radiate acoustic power. More specifically,

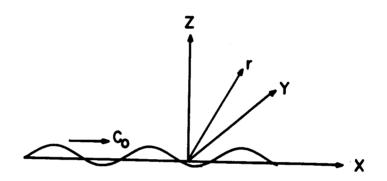


FIG. C.1. SKETCH OF A LINE MONOPOLE SOURCE.

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$$p(x,r,t) = constant \times \frac{1}{(k_r r)^{\frac{1}{2}}} e^{i(k_o x + k_r r - \omega_o t)}, \quad (C.6)$$

where the radial wavenumber ${\bf k}_{{\bf r}}$ is given by

$$k_{r} = (k_{a_{0}}^{2} - k_{0}^{2})^{\frac{1}{2}}$$
 (C.7)

Since $k_0^2 < k_{a_0}^2$ (see Eq. C.5), k_p is real and the sound is propagated radially outwards from the x-axis.

Now consider the alternate case (Case No. 2) where the source velocity $c_{\rm O}$ is subsonic, i.e.,

$$|c_0| < c$$
, (C.8)

or equivalently,

$$|k_0| > k_a_0. \tag{C.9}$$

For this case, the far field pressure p(x,r,t) is zero; in other words the line source cannot deliver any acoustic power. More specifically,

$$p(x,r,t,) = 0$$
 (C.10)

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This happens because the radial wavenumber k_r is imaginary,

$$k_r = +1 (k_0^2 - k_{a_0}^2)^{\frac{1}{2}}$$
, (C.11)

and the *near field* pressure decays exponentially in the radial direction.

Let us reconsider the above results in terms of the spatial Fourier transform $\tilde{q}(k_1,t)$ of Eq. (C.1). First, define the general Fourier transforms.

$$\tilde{q}(k_{1},t) = \frac{1}{2\pi} \int q(x,t) e^{-ik_{1}x} dx$$
 (C.12)

$$q(x,t) = \int \tilde{q}(k_1,t) e^{ik_1x} dk_1 \qquad (C.13)$$

Unless stated otherwise, the limits of integration are always to be taken from $-\infty$ to $+\infty$. For later use, the temporal Fourier transforms shall also be needed, defined as follows.

$$\tilde{q}(k_1,\omega) = \frac{1}{2\pi} \int \tilde{q}(k_1,t) e^{i\omega t} dt \qquad (C.14)$$

$$= \frac{1}{(2\pi)^2} \int \int q(x,t) e^{-i(k_1x-\omega t)} dxdt$$
 (C.15)

$$q(x,t) = \iint \tilde{q}(k_1,\omega) e^{i(k_1x-\omega t)} dk_1 d\omega \qquad (C.16)$$

$$= \int \tilde{q}(x,\omega) e^{i\omega t} d\omega \qquad (C.17)$$

Substituting q(x,t) of Eq. (C.1) into Eq. (C.12),

$$\tilde{q}(k_{1},t) = Q_{0}e^{-i\omega_{0}t} \delta(k_{1}-k_{0}), \qquad (C.18)$$

where δ is the Dirac delta function. Figure C.2 illustrates $\tilde{q}(k_1,t)$ for Case No. 1 (radiation) and for Case No. 2 (no radiation). The "radiation span" along the wavenumber k_1 is centered around the wavenumber $k_1 = 0$ and ranges from $-k_a$ to $+k_a$. This radiation span is shown in Fig. C.2 as a o shaded strip.

Let us reformulate the condition of no radiation in terms of velocities and Mach numbers. The two extremes $-k_a$ and $+k_a$ of the radiation span correspond respectively to the o ao lowest and the highest velocities that the source wave can have

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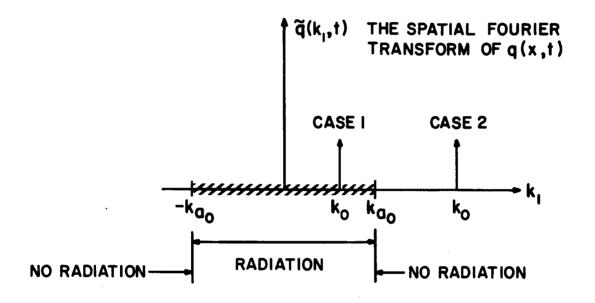


FIG. C.2. CASES OF RADIATION (No. 1) AND NO RADIATION (No. 2) ILLUSTRATED IN TERMS OF THE WAVENUMBER k_1 .

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for no radiation to occur (Case 2). These extreme velocities and Mach numbers are,

$$c_{\ell} = \frac{\omega_{0}}{-k_{a_{0}}} = -c$$

$$m_{\ell} = \frac{c_{\ell}}{c} = -1$$

$$c_{u} = \frac{\omega_{0}}{+k_{a_{0}}} = +c$$

$$m_{u} = \frac{c_{u}}{c} = +1$$
(C.20)

The source wave Mach number m_0 is defined as:

$$m_{o} = \frac{c_{o}}{c}$$
(C.21)

Thus, the condition of no radiation (Case 2) becomes

$$m_{\ell} < m_{O} < m_{U}$$
 (C.22)

Next, consider a spatially frozen but otherwise arbitrary pattern q(x,t) traveling, as before, with fixed velocity c_0 in the x-direction. Thus,

$$q(x,t) = Q(x-c_t)$$
 (C.23)

In contrast to Eq. (C.1), for which there was one wavenumber k_0 , one frequency ω_0 , and one velocity c_0 , for Eq. (C.23) there is a range of wavenumbers k_1 , a corresponding range of frequencies ω , and one velocity c_0 . Using Eq. (C.15), the Fourier transform of q(x,t) of Eq. (C.23) is

$$\tilde{q}(k_1, \omega) = \frac{1}{(2\pi)^2} \int \int Q(x-c_0 t) e^{-i(k_1x-\omega t)} dxdt$$
 (C.24)

$$= \frac{1}{2\pi} \int \tilde{Q}(k) e^{-ik_1 c_0 t + i\omega t} dt \qquad (C.25)$$

$$= \widetilde{Q}(k) \delta(\omega - k_1 c_0) , \qquad (C.26)$$

where

$$\tilde{Q}(k) = \frac{1}{2\pi} \int Q(x) e^{-ik_1 x} dx \qquad (C.27)$$

Figure C.3 shows the straight lines along which $\tilde{q}(k_1,\omega)$ of Eq. (C.26) is non-zero. Analogous to Fig. C.2, straight lines corresponding to Case 1 (radiation) and Case 2 (no radiation) are illustrated. Notice that as frequency ω increases, the radiation span $2k_a$ over wavenumber k_1 also increases linearly, But, as long as the source velocity magnitude $|c_0|$ is subsonic, there is no radiation to the far field. Incidentially, the upper right and the lower left quadrants of $\omega - k_1$ plane correspond to positive phase velocities, (i.e., velocities along increasing x), whereas the upper left and the lower right quadrants of $\omega - k_1$ plane correspond to negative phase velocities.

This completes the discussion of a continuous frozen pattern of line sources in a stationary acoustic medium. Consideration of the fact that the acoustic medium is moving uniformly, will simply alter the radiation span along k_1 , as will be discussed in Sec. C.2. However, a frozen convecting pattern along x_1 , will or will not radiate, by exactly analogous rules as developed here, i.e., in terms of the convection or phase velocity c_0 of the pattern.

Finally, note that for a continuous convecting line source of *finite* length, even if the convection velocity c_o is subsonic, there will be inevitable radiation from the two ends of the line source. For low enough frequencies, the two ends may be less than half an acoustic wavelength apart, in which case there may

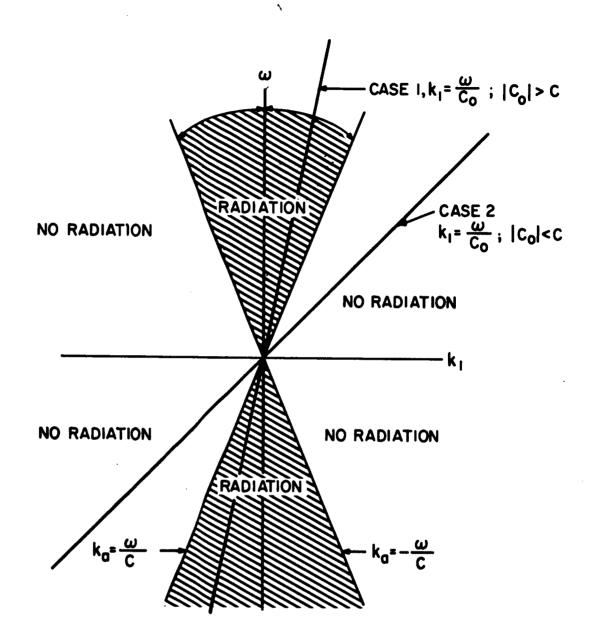


FIG. C.3. CASES OF RADIATION AND NO RADIATION FOR A SPATIALLY FROZEN ARBITRARY PATTERN, ILLUSTRATED IN THE $k_{_{\rm 2}}$, ω plane.

be partial cancellation from the two end sources. For higher frequencies, the two end sources will radiate independently. This last remark is discussed more fully below when discrete line sources are considered in a stationary acoustic medium. However, the SBLE is regarded as a continuous line array, and a typical rotor wake impinging on it has a *local* convection velocity co along the *span* of the SBLE. Thus the preceding discussion of continuous line sources, or rather its related extension in Sec. C.2, where account is taken also of the moving-medium acoustics, is applied to determine the SBLE sweep; the criterion that is applied is in terms of the spanwise local velocity of the rotor wake along the SBLE.

Discrete Line Source

Now consider an array of equispaced coherent monopoles, spaced d apart (see Fig. C.4), where:

$$x_{j} = d_{j}$$
, $j = 0, \pm 1, \pm 2...$ (C.28)

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In analogy with Eq. (C.1), consider one wavenumber k_0 , one frequency ω_0 and the corresponding phase velocity c_0 . Thus, the source number j has the strength $q(x_j,t)$ given by

$$q(x_{j},t) = Q_{0} e^{i(k_{0}x_{j}-\omega_{0}t)} \delta(x-d_{j})$$
 (C.29)

The entire source strength can be written as

$$q(x,t) = Q_0 \sum_{j=-\infty}^{+\infty} e^{i(k_0 x - \omega_0 t)} \delta(x-dj) , \qquad (C.30)$$

and the phase velocity $\boldsymbol{c}_{_{O}}^{},$ as before, is given by

$$c_{o} = \omega_{o}/k_{o} \qquad (C.31)$$

Eq. (C.12) is used to find the spatial Fourier transform of q(x,t) of Eq. (C.30),

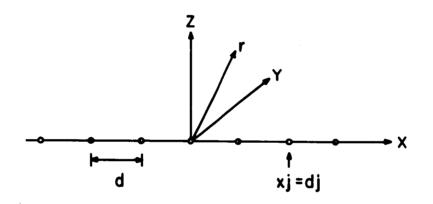


FIG. C.4. SKETCH OF AN ARRAY OF POINT SOURCES.

$$\tilde{q}(k_{1},t) = \frac{Q_{0}}{2\pi} \int e^{-ik_{1}x} e^{i(k_{0}x-\omega_{0}t)} \sum_{j=-\infty}^{+\infty} \delta(x-dj) dx$$
(C.32)

$$= \frac{Q_0}{2\pi} e^{-i\omega_0 t} \sum_{j=-\infty}^{+\infty} e^{-i(k_1-k_2)dj}$$
(C.33)

$$= \frac{Q_0}{d} e^{-i\omega_0 t} \sum_{m=-\infty}^{+\infty} \delta(k_0 - k_1 - \frac{2\pi m}{d}) \qquad (C.34)$$

 $\tilde{q}(k_1,t)$ thus consists of an infinite string of Dirac delta functions equispaced along the wavenumber k_1 , the spacing between two adjacent delta functions being $2\pi/d$. It is only the "fundamental mode" or harmonic at $k_1 = k_0$ (for m = 0 in Eq. C.34) that corresponds to the trace velocity c_0 of Eq. (C.31). The remaining infinite harmonics correspond to infinite other velocities. The rule of radiation (Case 1) or no radiation (Case 2) is exactly the same as the one developed for the continuous array and illustrated in Fig. (C.2). If the fundamental mode or any harmonic(s) lie within the radiation span $(-k_{a_0}, k_{a_0})$, radiation will occur from the fundamental mode or from the harmonic(s) lying within the radiation span.

However, a more interesting and new feature of the discrete array is the classification based on a different criterion. That classification is as follows:

Case A,
$$\frac{2\pi}{d}$$
 > 2k_a, or d < $\frac{\lambda_a}{2}$ (C.35)

Case B,
$$\frac{2\pi}{d} < 2k_{a_0}$$
, or $d > \frac{\lambda_{a_0}}{2}$ (C.36)

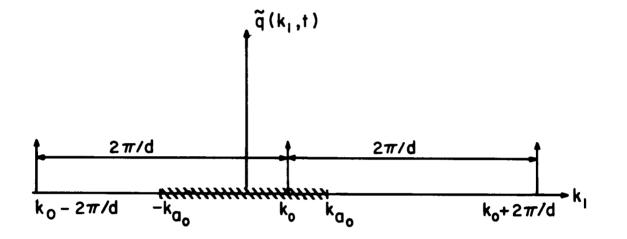
For Case A, the spacing $2\pi/d$ along wavenumber k, between harmonics is greater than the radiation span $2k_a$, since $k_{a_0} = 2\pi/\lambda_a$, where λ_a^{i} is the acoustic wavelength at frequency ω_0 , ω_0 , ω_0 the same condition is expressed by the statement that the array spacing d is smaller than half the acoustic wavelength. For Case B, the opposite situations occur in the wavenumber and spatial descriptions.

The importance of these two cases is depicted in the next few figures. Figure C.5 describes Case Al (the numbers 1 and 2 denote the older classification, 1 corresponds to radiation occurring, 2 corresponds to radiation not occurring). The radiation occurs from the fundamental mode at $k_1 = k_0$, but since $2\pi/d > 2k_a$, no other harmonic can radiate. Figure C.6 also describes ^O Case Al, however, this time a harmonic, and *only one* harmonic radiates. Figure C.7 shows the Case A2, a situation one would hope to achieve. The fundamental mode at $k_1 = k_0$ lies just outside the radiation span, and no harmonic lies within the radiation span, hence no radiation occurs. Note that for this desired situation, the constraint of Eq. (C.8) (or equivalently of Eq. C.22) as well as the constraint of Eq. (C.35) applies.

Finally, Fig. C.8 shows Case B1. There is no Case B2. Radiation must occur through same mode(s), whether the phase velocity c_0 is subsonic or supersonic. In other words, arranging for Case B, i.e., having array spacing d greater than half the wavelength, is basically a poor design.

Note that in contrast to the continuous line array for which the discussion related to Fig. C.2 for frequency ω_0 could be generalized to discussion related to Fig. C.3 for all frequencies, the discussion of the discrete array presented above *cannot* be similarly generalized to all frequencies. This is because the array spacing d is in general fixed, whereas the radiation span 2k increases linearly with increasing ω_0 . Thus, Case A for $^{\circ}$ frequency ω_0 is bound to merge into Case B at some higher frequency.

Aside from the re-definition in Sec. C.2 of the radiation span in wavenumber k_1 , induced by consideration of moving-medium acoustics, the above discussion of a discrete array is applied to determine the number of stator blades, the spacing d corresponding to the circumferential spacing between two adjacent stator tips, and frequency ω_0 corresponding to the blade passage frequency.



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FIG. C.5. CASE A1 FOR A DISCRETE ARRAY; RADIATION FROM THE FUNDAMENTAL HARMONIC AT k_0 .

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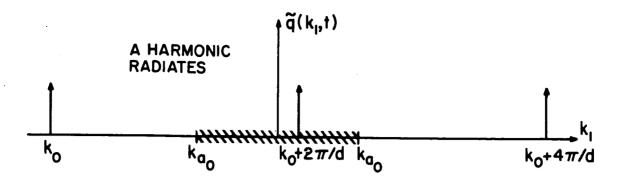


FIG. C.6. CASE A1; RADIATION FROM A HARMONIC OTHER THAN THE FUNDAMENTAL.

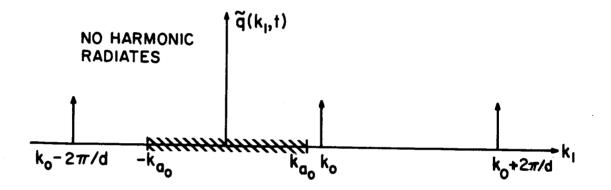
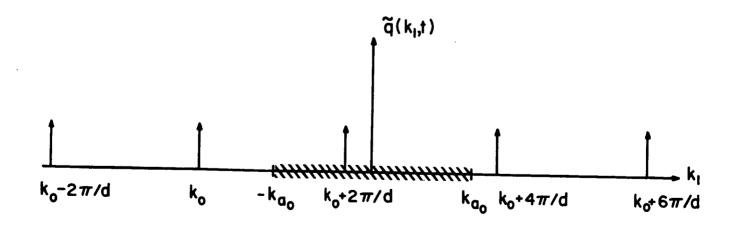


FIG. C.7. CASE A2; NO RADIATION.



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FIG. C.8. CASE B1; INEVITABLE RADIATION.

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C.2 Acoustics of a Moving Medium

The only task that needs to be performed in this section is to investigate how the radiation span $(-k_a, +k_a)$ along the wavenumber k, gets modified due to the ^o fact that the acoustic medium is moving uniformly with subsonic velocity $\underline{u} = (u_1, u_2, u_3)$, where u_1, u_2, u_3 are the velocity components in the x, y and z directions.

Once again, consider the line monopole source of Fig. C.l, with Eqs. (C.l) through (C.5) still applicable. In addition to the radial coordinate r of Eq. (C.3), the corresponding radial vector \underline{r} is defined as

r = (y, z) (C.37)

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The far field pressure P(x,r,t) now must satisfy the following field equation, (Morse and Ingard, 1964),

$$\nabla_{p}^{2} + k_{a_{0}}^{2} \left[1 + \frac{1}{\omega_{0}} \left(u_{1} \frac{\partial}{\partial x} + u_{2} \frac{\partial}{\partial y} + u_{3} \frac{\partial}{\partial z}\right)\right]^{2} p = 0 \quad (C.38)$$

For $u_1 = u_2 = u_3 = 0$, Eq. (C.38), reduces to the usual Helmholtz equation applicable for a stationary acoustic medium. In analogy with Eq. (C.11), a criterion is needed that the radial wavenumber k_r must satisfy for no radiation to occur to the far field (i.e., Case 2). However, in analogy with generalization of Eq. (C.3) to Eq. (C.37), a radial wavenumber vector k_r is defined as

$$k_{\underline{r}} = (k_2, k_3),$$
 (C.39)

where k_2 and k_3 are the wavenumber components of the radially outwards propagating wave.

Now the important phase aspect of the far field pressure P(x,r,t) (for a given value of r in the farfield) is given by the correspondingly generalized version of Eq. (C.6)

$$P(x,\underline{r},t) = \text{constant} \times e^{i(k_0x - \frac{k_r}{r} \cdot \underline{r} - \omega_0 t)}, \qquad (C.40)$$

where

$$\frac{\mathbf{k}_{\mathbf{r}}}{\mathbf{r}} \cdot \mathbf{r} = \mathbf{k}_{2}\mathbf{y} + \mathbf{k}_{3}\mathbf{z} \tag{C.41}$$

The following definitions are required:

$$k_{r} = \left|\frac{k_{r}}{2}\right| = \left(k_{2}^{2} + k_{3}^{2}\right)^{\frac{1}{2}}$$
 (C.42)

$$u_{r} = |\underline{u}_{r}| = (u_{2}^{2} + u_{3}^{2})^{\frac{1}{2}}$$
 (C.43)

$$k_{2} = k_{r} \sin \alpha_{k} , \qquad (C.44)$$

$$k_{3} = k_{r} \cos \alpha_{k} , \qquad (C.45)$$

$$u_{2} = u_{r} \sin \alpha_{u} , \qquad (C.46)$$

$$u_{3} = u_{r} \cos \alpha_{u} , \qquad (C.47)$$

so that

$$\frac{k_{r}}{r} \cdot \frac{u_{r}}{r} = k_{2} u_{2} + k_{3} u_{3} = k_{r} u_{r} \cos \alpha , \qquad (C.48)$$

where

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$$\alpha = (\alpha_k - \alpha_u) \quad . \tag{C.49}$$

Thus, α is the angle between the radial wavenumber vector k_r (or the radial location vector (y,z) of observation in the farfield) and the radial flow vector u_r .

Now, substituting Eq. (C.40) into Eq. (C.38) gives the following relation,

C-17

$$-k_{r}^{2} - k_{0}^{2} + k_{a_{0}}^{2} \left[1 - \frac{1}{\omega_{0}} \left(u_{1}k_{0} + u_{r}k_{r}\cos\alpha\right)\right]^{2} = 0 \qquad (C.50)$$

which can be rewritten as a quadratic equation in ${\bf k}_{\rm r}$ as follows

$$Ak_{r}^{2} + Bk_{r} + C = 0$$
 (C.51)

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where

$$A = \frac{k_{a_{o}}^{2}}{\omega_{o}^{2}} u_{r}^{2} \cos^{2}\alpha - 1 = m_{r}^{2} \cos^{2}\alpha - 1 \qquad (C.52)$$

$$B = 2 \frac{k_{a_{o}}^{2}}{\omega_{o}^{2}} u_{1}k_{o}u_{r} \cos\alpha - 2 \frac{k_{a_{o}}^{2}}{\omega_{o}} u_{r}\cos\alpha$$

$$= 2(m_{1}m_{r}k_{o}\cos\alpha - k_{a_{o}}m_{r}\cos\alpha) \qquad (C.53)$$

and

$$C = (-k_0^2 + k_a_0^2 - 2 \frac{k_a^2}{\omega_0} u_1 k_0 + \frac{k_a^2 u_1^2}{\omega_0^2} k_0^2)$$

$$= (-k_0^2 + k_a_0^2 - 2k_a_0^m k_0 + m_1^2 k_0^2)$$
(C.54)

In the above equations appropriate Mach numbers are introduced by division of velocities by the sound speed c = ω_0/k_{a_0} .

For the stationary acoustic medium, the condition on radial wavenumber magnitude k_r , for no radiation to occur (Case 2), was that k_r be imaginary (see Eq. C.11). In analogy with that requirement for no radiation to occur, it is required that k_r of Eq. (C.51) be complex (with positive imaginary part). That will happen if and only if

$$AC - B^2/4 > 0$$
 . (C.55)

Now, the left hand side of Eq. (C.55) does not contain k_r , but is a quadratic form in k_o , the wavenumber of the source wave. Thus, Eq. (C.55) can be rewritten as

$$DK_{0}^{2} + Ek_{0} + F > 0$$
, (C.56)

where

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$$D = 1 - m_1^2 - m_r^2 \cos^2 \alpha$$
 (C.57)

$$E = 2k_{a_0} m_1$$
 (C.58)

$$F = -k_{a_0}^2$$
(C.59)

The minimum value of D occurs for $\alpha = 0$ or π (i.e., when k_r and u_r are coincident or oppositely directed. This minimum $\frac{1}{r}$ value D_{minm} is given by

$$D_{minm} = 1 - m_1^2 - m_r^2 = 1 - m^2 > 0$$
,

where m is total flow Mach number. Since $D_{\min m}$, and therefore D is always positive, the left hand side of Eq. (C.56) is positive for large $|k_0|$ (i.e., for $k_0 + \pm \infty$), being dominated by the first term Dk_0^2 . This behavior, incidently, is consistent with the inequality expressed by Eq. (C.56). Recalling that the inequality of Eq. (C.56) is the condition on k_0 for no radiation to occur (i.e., Case 2), the radiation will, in fact, take place for a range of wavenumbers k_0 of relatively small magnitude. This range, the radiation span along axial wavenumber k_1 , is determined by the two real roots k_a and k_a of the quadratic form of Eq. (C.56)

$$k_{a_{0\pm}} = k_{a_{0}} \frac{-m_{1}^{\pm}(1-m_{r}^{2}\cos^{2}\alpha)^{1/2}}{(1-m_{1}^{2}-m_{r}^{2}\cos^{2}\alpha)}$$
(C.60)

In analogy with Fig. C.2, this radiation span is shown as a shaded strip in Fig. C.9. The center O' of the span is shifted to the left by the amount $k_a m_1/D$ (with D given by Eq. (C.57) from the origin O (k = 0). O This shift resulting from first (common) term on the right hand side of Eq. (C.60), is interpreted as a Galilean shift. The equal intervals (k , O') and O', k), resulting from the second terms on the right o hand side of o Eq. (C.60) are interpreted as Lorentz half-spans.

Also shown in Fig. C.9 is $\tilde{q}(k_1,t)$ for a phase wave whose phase velocity c₀ is supersonic in the fixed coordinate system (or equivalently whose k_0 is less than k_a), yet since k_0 lies outside the radiation span, the particular phase wave illustrated will not radiate to the farfield.

Reformulation of the condition of no radiation in terms of Mach numbers can be done along exactly similar lines as done in Eqs. (C.19), (C.20) and (C.21). Thus, the upper and lower permissible Mach numbers m_{11} and m_{22} are given by

$$m_{\ell} = \frac{(1-m_{r}^{2}-m_{r}^{2}\cos^{2}\alpha)}{-m_{r}-(1-m_{r}^{2}\cos^{2}\alpha)^{1/2}} = m_{1}-(1-m_{r}^{2}\cos^{2}\alpha)^{1/2} , \quad (C.61)$$

$$m_{u} = \frac{(1-m_{1}^{2}-m_{r}^{2}\cos^{2}\alpha)}{-m_{1}^{2}+(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}} = m_{1}^{2}+(1-m_{r}^{2}\cos^{2}\alpha)^{\frac{1}{2}}, \quad (C.62)$$

and for no radiation to occur, the following must be satisfied:

$$m_{\ell} < m_{o} < m_{u}$$
 (C.63)

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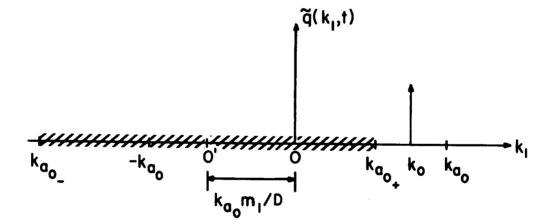


FIG. C.9. SKETCH OF RADIATION SPAN ALONG WAVENUMBER k, FOR A MOVING ACOUSTIC MEDIUM.

Extension similar to that from Figs. C.2 to C.3 can also be easily performed for the present case; as a result of the medium motion, the acoustic "cones" of radiation in the $\omega-k_1$ plane will be asymmetrical about the ω axis.

Finally, the entire discussion of discrete arrays in Sec. C.2 can be applied here with the newly defined radiation span.

C.3 An Estimate Of Overall Power Radiated From The Stator

Figure C.10 shows the wake velocity deficits as viewed in time at one SBLE near the tip. There are f such deficits per second, where f = 8600 Hz is the rotor blade passage frequency. Each individual^r wake deficit, v(t) has an approximately Gaussian shape around its peak deficit value v, thus

$$v(t) \approx v_0 e^{-t^2/2T^2}$$
 (C.64)

where the "standard deviation" T, and the maximum deficit $v_{_{\rm O}}$ are estimated to be:

$$\Gamma \approx 9.6 \times 10^{-6}$$
 sec. (C.65)

$$v_o \approx 44 \text{ m/sec} (144 \text{ ft/sec})$$
 (C.66)

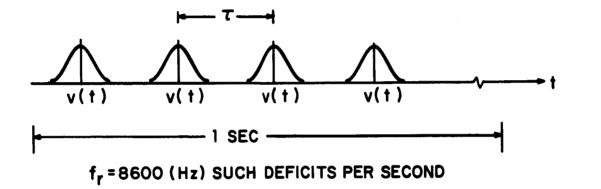
The maximum deficit, v corresponds to 10° change in angle of attack. The time interval τ between consecutive deficits is given by

$$\tau = \frac{1}{f_r} \approx 1.16 \times 10^{-4} \text{ sec.}$$
(C.67)

Note that τ is about an order of magnitude greater than T, in other words the wake deficits are narrow in time when compared to their rate of arrival.

The above data regarding the wake velocity deficits was developed from Kemp and Sears (Ref. 18). The description of the wake velocity deficit in a spatial coordinate, x, can be obtained by assuming that the wakes arrive at the SBLE tips as (locally) frozen spatial patterns, being convected along with the local gas speed V, where

$$V = (m_c^2 + m_a^2)^{1/2} c \approx 195 \text{ m/s } (641 \text{ (C.68)} \text{ ft/sec})$$



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FIG. C.10. SKETCH OF TIME HISTORY OF WAKE VELOCITY DEFICITS AS THEY IMPINGE ON A SINGLE SBLE TIP.

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 $\rm m_{c}$ and $\rm m_{a}$ being given by Eqs. 21 and 22 . Thus, the spatial picture of a wake velocity deficit is obtained by the transformation,

$$x = Ut$$
 . (C.69)

For estimating the overall acoustic power radiated from the stator it is convenient, as shown below, to integrate the results in the time domain. However, in order to get a qualitative understanding of the situation in the frequency domain we discuss briefly the Fourier transform $\tilde{v}(\omega)$ of v(t) of Eq. (C.64).

$$\tilde{v}(\omega) = \frac{1}{2\pi} \int v(t)e^{i\omega t} dt$$
 (C.70)

$$= \frac{\mathrm{Tv}_{0}}{(2\pi)^{1/2}} e^{-\omega^{2} \mathrm{T}^{2}/2}$$
(C.71)

The Fourier transform $\tilde{v}^{\,\prime}\,(\omega)$ of the sequence of pulses of Fig. C.10 may then be written as

$$\widetilde{v}'(\omega) = \frac{1}{2\pi} \int v'(t) e^{i\omega t} dt$$
$$= \frac{1}{2\pi} \int \sum_{j=-\infty}^{+\infty} v(t-\tau,j) e^{i\omega t} dt \qquad (C.72)$$

$$= \tilde{v}(\omega) \omega_{r} \sum_{n=-\infty}^{+\infty} \delta(\omega - n\omega_{r}) , \qquad (C.73)$$

where ω is the blade passage frequency in radians/sec (see Eqs. 0 17 and 18),

$$\omega_{0} = 2\pi f_{r} = \frac{2\pi}{\tau} = \Omega B$$
 (C.74)

Thus, as expected, the frequency content of the rotor wake velocity deficits v'(t) consists of the various rotor harmonics n. Since $\tilde{v}(\omega)$ does not decay appreciably with increasing frequency (the "standard deviation" of $\tilde{v}(\omega)$ is 1/T) the higher rotor harmonics at $n = \pm 2$, ± 3 , etc. have nearly the same strength or amplitude as the fundamental harmonic at $n = \pm 1$.

Now, reverting back to the time domain analysis, the fluctuating lift l(t) generated at the leading edge of a SBLE tip from impingement of one wake deficit v(t) is given by

$$l(t) = \int_{0}^{\infty} v(t') h(t-t') dt'$$
, (C.75)

where h(t) is the unit impulse response function derived from Küssner's function [Bisplinghoff *et al.*, Ref. 19]. Since the essential uncancelled fluctuating lift is restricted to a relatively small segment of the SBLE span near the tip, use of Küssner's function, valid for low aspect ratio, is readily justified for the present calculation. Since Küssner's function gives the lift due to a sharp-edged gust (i.e., due to a gust that is a unit step function), the unit impulse response function h(t) is obtained by differentiating the Küssner's function. h(t), so obtained, is given by

$$h(t) = C_{L} \left[\frac{0.13}{2\tau_{b}} e^{-0.13t/\tau_{b}} + \frac{1}{2\tau_{b}} e^{-t/\tau_{b}} \right], \qquad (C.76)$$

where $\tau_{\rm b}$ is the time taken by the gust to travel the (swept) semichord b.

$$\tau_{\rm b} = b/U \approx 1.33 \text{ x } 10^{-4} \text{ sec.}$$
 (C.77)

The above estimate of τ_B is based on b ≈ 0.034 m (0.11 ft), and U of Eq. C.68. The lift coefficient C_T is given by

$$C_{\rm L} \approx 2\pi\rho Ub \frac{\lambda_{\rm r}}{4}$$
, (C.78)

where ρ is the medium density (2.4×10^{-3} lb-sec²/ft^{*} $\approx 1.24 \text{ Kg/m}^3$) and λ_r is the acoustic wavelength at blade passage frequency f_r (Eq. 26). Note that in Eq. C.78, $\lambda_r/4$ denotes a rough estimate of the SBLE span near the tip from which the uncancelled fluctuating lift is estimated to radiate. Now, this choice of $\lambda_r/4$ is suitable (only) for the rotor fundamental harmonic at frequency ω_r . For the higher rotor harmonics of Eq. C.73, correspondingly smaller spanwise length scales would be more appropriate. However, in the time domain analysis that is being pursued, the choice of $\lambda_r/4$ in Eq. C.78 is taken to apply to all the rotor harmonics, therefore the resulting estimate of the overall (i.e., frequency-integrated) radiated power is liable to be conservative.

Substituting Eqs. (C.64) and (C.76) into Eq. (C.75), enables calculation of fluctuating lift l(t) at a single SBLE tip due to the impingement of a single wake velocity deficit v(t). Since the minimum time constant τ_b of h(t) is much larger than the time constant or "standard" deviation" T of v(t) [compare Eqs. (C.65) and (C.77)], for evaluating l(t) from Eq. (C.75), one can justifiably approximate v(t) of Eq. (C.64) as follows,

$$v(t) \approx v_{\rho} (2\pi)^{1/2} T\delta(t)$$
 (C.79)

The constant $(2\pi)^{1/2}$ T in Eq. (C.79) is introduced so as to make the total "area" (in other words, the integral $\int v(t) dt$ the same for Eqs. (C.64) and (C.79). From Eq. (C.70) and (C.71), note that this area is equal to $2\pi\tilde{v}(\omega)|_{\omega=0}$. Substituting Eq. (C.79) into Eq. (C.75), gives

$$\ell(t) = v_0 (2\pi)^{1/2} T h(t) . \qquad (C.80)$$

From the point of view of generation of *steady* lift, the stator blade chord is expected to be oriented parallel to the flow velocity U at its leading edge, so that a zero *mean* angle of attack is ensured. Hence, the wake-deficit-induced fluctuating lift of Eq. C.80 is oriented normal to the flow velocity U. The acoustic intensity I(t) radiated by this "transverse" dipole (i.e., the direction of fluctuating lift is normal to flow) is given by [Lighthill,Ref. 15; Morse and Ingard, Ref. 17],

$$I(t) = [2\dot{\ell}(t)]^{2} \frac{1}{12\rho\pi c^{3}} G_{2}(m) , \qquad (C.81)$$

where l(t) = d/dt l(t) and $G_2(m)$ is a function of the flow Mach number m = U/c,

$$G_{2}(m) = \frac{3}{4} \left[\frac{2}{m^{2}(1-m^{2})} - \frac{1}{m^{3}} \ell n \frac{1+m}{1-m} \right].$$
 (c.82)

The factor 2 appearing with $\ell(t)$ in Eq. C.81 accounts for the baffling effect due to the presence of the duct wall (assumed to be acoustically rigid) enclosing the SBLE tip.

The radiated acoustic energy E, associated with the intensity I(t) of Eq. C.81 is given by

$$E = \int_{0}^{\infty} I(t) dt. \qquad (C.83)$$

From Eqs.C .76, C.80 and C.81, we note that the only time dependent factor of I(t) involves h(t) of Eq. C.76, hence, the integral to be evaluated is

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$$\int_{0}^{\infty} \dot{h}(t)^{2} dt = C_{L}^{2} \frac{0.133}{\tau_{b}^{3}} , \qquad (C.84)$$

thus, substituting Eqs. C.80, C.81 and C.84 into Eq. C.83 we get

$$E = \frac{2}{3} \frac{G_2(m)}{\rho c^3} (v_0 T C_L)^2 \frac{0.133}{\tau_b^3}$$
(C.85)

Now, E is the acoustic energy radiated from a single SBLE tip due to impingement of a single wake velocity deficit v(t). Hence, the acoustic power radiated from a single SBLE tip is Ef, where f is the rate of impingement of wake deficits on the SBLE tip.

Next, assume that the V individual SBLE tips radiate more or less incoherently (an assumption particularly valid for higher rotor harmonics n of Eq. C.73). Hence, the power radiated from the V stator tip sources is $\text{Ef}_n V$.

Finally, even though the calculation for the power radiated from the stator leading edge sources at the hub is not carried out separately, because of closer circumferential spacing between these hub sources, the total power radiated from the stator hub is likely to be considerably less than that from the stator tip. The total power I radiated from the stator is conservatively estimated to be given by

$$\Pi = 2Ef_{r}V \qquad (C.86)$$

Substituting in Eqs. (C.85) and (C.86) the numerical values quoted for various parameters (with V = 59) gives

 $\Pi = 130.5 \text{ dB re } 10^{-12} \text{ watt}$ (C.87)

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APPENDIX D

NOTES ON EMPIRICAL CALCULATION OF FAN NOISE LEVELS

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APPENDIX D: NOTES ON EMPIRICAL CALCULATION OF FAN NOISE LEVELS

As mentioned in Section 7, all components of the rotor and stator noise spectrum could not be calculated from basic considerations. The previous Appendix gives a noise level calculation for the residual stator noise sources. In the interest of determining what reduction in levels the swept rotor might be expected to cause, Burdsall's empirical correlation (Ref. 19), was exercised for both the actual fan model and a "full scale" counterpart. The parameters required in Burdsall's routine are given in Table D-1. Figure D-1 summarizes the narrow band power levels and spectra for the various components. (SPL arbitrarily computed at 150 ft., 60° from rotor axis) showing the predominance of MPT'S. Of course, the details of the MPT spectrum vary from fan to fan due to their origin in manufacturing tolerances. Fig. D-2 shows a typical comparison of Burdsall's prediction with measured data, indicating a fairly large fluctuation in harmonic levels around the mean line of the prediction. In Fig. D-1, it is clear that according to this scheme, elimination of MPT's would reduce the tone levels considerably. However, note that in Fig. D-1, the line is an envelope of discrete frequency levels at various multiples of rotation rate while the broadband spectrum is continous. Thus, integration into constant percentage bandwidths, and into overall levels will lead to the MPT and broadband levels being very nearly identical.

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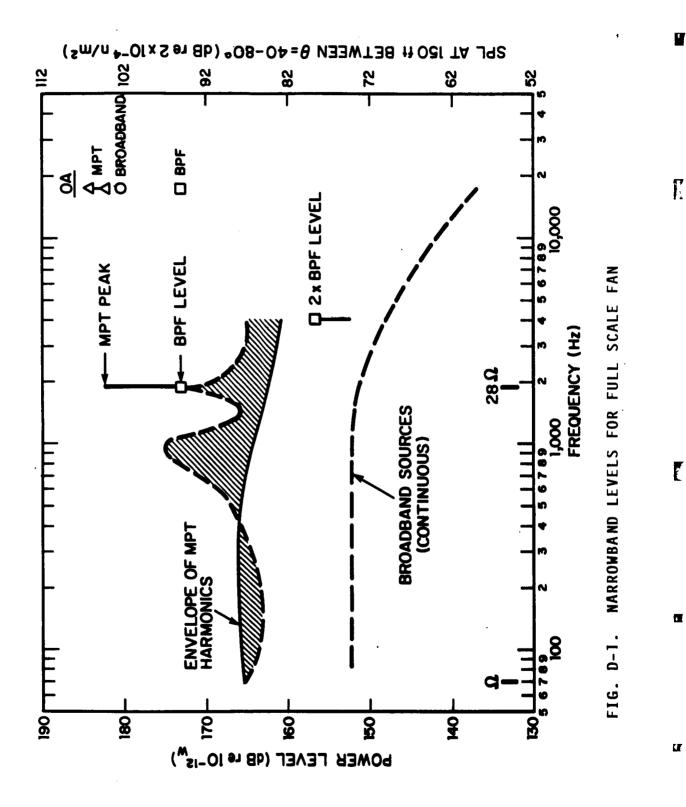
As a final point, it is interesting to note that the power level of the BPF tone (non-MPT noise) is ~10 dB above the predicted level for the swept stator as described in Appendix C.

			• • •	• •	DISCRETE	COMBINATION
DESCRIPTION	UNITS	MODEL FAN	FULL SCALE	BROADBAND	TONE	TONE
fan tip diameter	inches	20.0	. 89.0	×	×	×
fan hub diameter	inches	9.5	42.0		_	×
number of blades	1	28.0	28.0	•••		×
number of vanes		39.0	59.0			
bypass ratio	· •	8.0	8.0	×		
rotor/stator space	inches '	3.94	, 17.6			
blade tip chord	inches	3.15	14.0			
fan tip dia. gradient	3 6 8 8	-0.222	-0.222			×
fan hub dia. gradient	* 7 *	0.444	0.444			×
blade suction surface radius of curvature	Inches	12.0 (avg.)	54.0			*
standard deviation of blade tip metal angle	radians	0.015	0.015			*
	fect	1.5	. 6.7		×	
radius to observer	feet	50.0	150.0	×	×	
fan speed	грт	18450	4130.0			
specific airflow rate	lb/sec/ft ²	32.8	32.0		-	
tip axial Mach number	8	0.62	0.62			*
fan pressure ratio		1.6	1.6		×	
diffusion factor (avg)	1	0.4	۳.0	×		
ambient inlet temp- temperature		46	46.0			
ambient inlet pressure	in. Hg	29.92	29.92			

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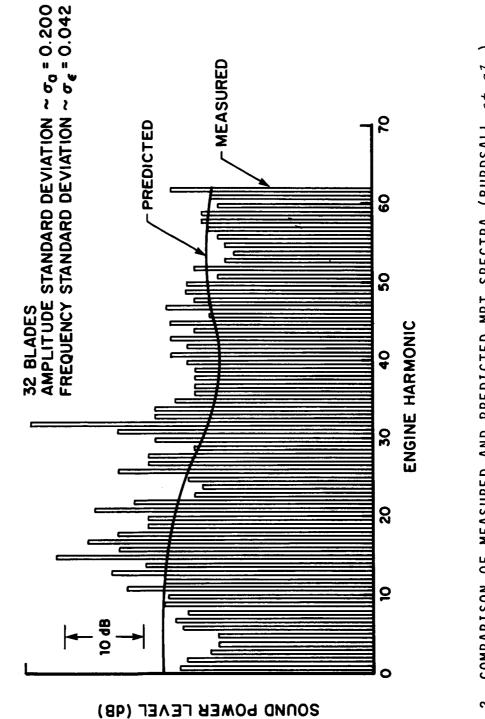
INPUT PARAMETERS FOR EMPIRICAL NOISE PREDICTION TABLE D-1

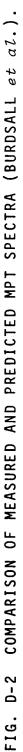
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APPENDIX E

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ALGORITHM FOR DERIVATION OF STATOR LEADING EDGE TRACE VELOCITY IN STATOR FIXED COORDINATES

APPENDIX E. ALGORITHM FOR DERIVATION OF STATOR LEADING EDGE TRACE VELOCITY IN STATOR FIXED COORDINATES

The geometry of a rotor wake as it reaches the stator is given in Fig. E-1. Refer to Fig. 9b in the text for 3-dimensional representation. The following four steps give the rotor wake shape and local trace velocity for both an unswept stator (or at the inlet plane of a swept stator), and for swept stators. Aerodynamic reaction on the rotor path by the stator is not taken into account.

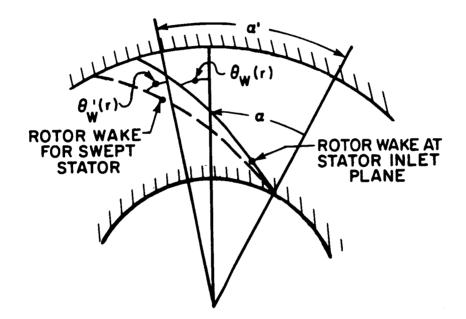


FIG. E-1. GEOMETRY FOR CALCULATION OF ROTOR WAKE SHAPE AND TRACE SPEED ON STATOR VANES.

1) Unswept, Unskewed Stator

The skew of the rotor wake at stator plane is $\alpha(r)$. The local angle between the wake and the radial direction may be derived from:

$$\tan \theta_{W} = r \frac{\delta \alpha}{\delta r}$$

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The trace velocity in the radial and axial directions is respectively

$$V_{T_{B_R}}(r) = (wr)/tan_w(r)$$

and

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where:

V = the trace velocity in blade fixed coordinates for radial and axial directions respectively.

2) Add Sweep $\mu_{\mbox{B}}$

The rotor wake in the conical surface of the swept leading edge is changed, as follows.

$$V_{T_{B_{R}}}(r) = \frac{wr}{\tan\theta'_{w}(r)}$$
 where $\tan\theta'_{w} = r \frac{\delta\alpha}{\delta r'}$

and

$$V_{T_{B_{x}}}^{\prime}(r) = V_{T_{B_{R}}}$$
 tan $\mu_{B}(r)$ where $\mu_{B}(r) = 1$ ocal blade sweep angle.

$$\alpha'(\mathbf{r}) = \alpha(\mathbf{r}) + \Delta x \frac{\delta \alpha(\mathbf{m})}{\delta x}$$
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where Δx is the downstream displacement of the leading edge caused by sweep.

$$\tan\theta'_{W}(\mathbf{r}) = r \frac{\delta\left[\alpha(\mathbf{r}) + \Delta x \frac{\delta\alpha(\mathbf{r})}{\delta \mathbf{r}}\right]}{\delta \mathbf{r}}$$
$$= r \frac{\delta\alpha(\mathbf{r})}{\delta \mathbf{r}} + r \Delta x \frac{\delta^{2}\alpha(\mathbf{r})}{\delta \mathbf{x}\delta \mathbf{r}} + r \frac{\delta\Delta x}{\delta \mathbf{r}} \frac{\delta\alpha(\mathbf{r})}{\delta \mathbf{x}}$$
$$= \tan\theta_{W} + \Delta x \frac{\delta\tan\theta_{W}}{\delta \mathbf{x}} + r \frac{\delta\alpha(\mathbf{r})}{\delta \mathbf{x}} \tan\mu_{B}$$
$$\tan\theta'_{W}(\mathbf{r}) = \tan\theta_{W} + r \frac{\delta\alpha(\mathbf{r})}{\delta \mathbf{x}} \tan\mu_{B} \qquad .$$

3) Trace Velocity for Swept Stator in Stator-Fixed Coordinates

The radial component of trace velocity is

$$V_{T_{B_{R}}}(r) = \frac{wr}{\tan\theta_{w}^{*} + r\frac{\delta\alpha(r)}{\delta x} \tan\mu_{B}}$$

where $\tan \theta_{W}^{*}(\mathbf{r}, \mathbf{x}) = \tan \theta_{W} + \Delta \mathbf{x} \frac{\delta \tan W}{\delta \mathbf{x}}$

The axial component is:

$$V_{T_{B_{x}}}(r) = \frac{wr \tan \theta_{B}}{\tan \theta_{W}^{*} + r \frac{\delta \alpha(r)}{\delta x} \tan \theta_{B}}$$

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where θ_W^* is determined from cross-plots of the wake path in the r, α plane at various axial locations, and $\delta \alpha / \delta r =$ the local wake helix pitch angle determined from wake crossplots in the meridional plane.

4) Transformation of Trace Velocity Amplitude From Stator-Fixed Coordinates to Gas-Fixed Coordinates

$$\begin{bmatrix} v_{T_{B_{R}},x} & v_{T_{G}} \end{bmatrix} = \begin{bmatrix} \left(v_{T_{B_{x}}} & v_{B_{x}} \right)^{2} + v_{T_{B_{R}}}^{2} + v_{G_{C}}^{2} \end{bmatrix}^{\frac{1}{2}}$$

where $V_{G_{x,c}}$ is the gas velocity in the axial and circumferential x,c directions, respectively.

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LIST OF SYMBOLS

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A	2	axial
A	=	inlet area (to rotor passage)
A _{min}	=	geometric throat area
As	=	area to choke the flow
$a_0 t/\lambda$	=	normalized distance
b	=	blade semichord
В	=	number of blades
с	=	chord length; sound speed
C ₀	=	phase or trace velocity; sound speed
CG	=	center of gravity
CL	=	lift coefficient
D	=	diffusion factor
DCA	=	double circular area (blade profile)
đ	=	distance from blade section c.g. to pressure surface; circumferential spacing between adjacent stator tips
dl	=	swept leading edge element
E	1	acoustic energy (radiated from a single SBLE tip)
f	=	Mach factor (= $1/M_{W_1L}$); frequency
fr	=	blade passage frequency of rotor blade

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ΔFj	=	centrifugal force at center of a blade volume element located at j
G ₂ (m)	=	function of flow Mach number
h	=	unit impulse response function
I	=	acoustic intensity
j	=	source number
k	=	wavenumber; constant
k _r	=	radial wavenumber
L	=	harmonic
L	=	distance from section c.g. to leading edge; fluctuating lift
LCF	=	low cycle fatigue
LE	=	leading edge
L _{mn}	=	rotor/stator interaction harmonic
m	=	circumferential mode number; component of Mach number; function in deviation angle formula
М	=	Mach number; moment
Mw	=	relative flow Mach number component
M _{w1}	= `	relative inlet Mach number
M ' W 1	=	Mach number required for a subsonic edge to achieve sonic (= M_{W_1})
M _{w1L}	=	component of Mach number which is always normal to the leading edge (= 1 for sonic LE; < 1 for subsonic LE

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^M w'L	=	Mach number required to make a subsonic LE a sonic LE
∆M _{ij}	=	moment of j force about c.g. of section i
N	=	rotation rate
n	=	shape parameter for polynomial blade forms; harmonic number
P	=	total pressure; static pressur
P or p	=	far field acoustic pressure
P	=	location of leading edge point
P/P	=	pressure ratio
Ŷ	=	static pressure after normal shock
PNL	=	Perceived Noise Level
q	=	source strength
Α	=	monopole source strength per unit length
R	=	distance from origin
R _c	=	radius of curvature of streamline
r	=	radial distance
S	=	circumferential blade spacing
SAP	=	Structural Analysis Program
SCF	=	Stress Concentration Factor
SBLE	=	Stator Blade Leading Edge
SR	=	sweep reversal

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t	=	thickness; time	
t(x)	=	thickness distribution	
т	=	standard deviation (time)	
TE	=	trailing edge	E
U or u	=	air velocity; wheel speed	
v	=	velocity	
v _o	=	peak velocity defect	
v	=	number of stator vanes; velocity	
۵Vj	=	volume element of a blade	
W	=	mass flow rate; velocity; inlet relative velocity	
\overline{W}	=	average velocity	
W _{as}	=	specific mass flow	E
w/w	=	flow deceleration rate	
W	=	relative velocity	
$\Delta \mathbf{x}$	=	downstream displacement of SBLE caused by sweep	
x	=	linear distance	
z, z _L	.=	axial coordinate of leading edge	

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GREEK

α	=	angle of attack; angle between radial wave vector k and radial flow vector u _r ; local Mach angle; wake displacement angle from radial
α ₃	=	stator inlet angle
β	=	relative flow angle; exit angles of flow
γ	=	setting angle; ratio of specific heats
δ	=	flow deviation angles; Dirac delta function
δ ₃	=	stator deviation angle
ε	=	slope of the relative flow velocity; slope between lines connecting section LE and CG and a line connecting LE with lower surfaces coordinate at mid-chord ($\varepsilon_{w_1} = \sin^{-1}V_r/w_1$)
εj	=	centerline-projected displacement of the c.g.'s of an airfoil section at r_j relative to one at r_i
λ	=	acoustic wavelength
λ	=	acoustic wavelength at blade passage frequency
η	=	polytropic state efficiency
θ	=	circumferential angle
θ _w	=	angle between radius and rotor wake centerline, unswept stator
θ _w '	=	angle between radius and rotor wake centerline, swept stator

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θL	=	angular abscissa of leading edge point
μ	=	Mach cone angle; radial order of acoustic modes
μ _B	=	stator sweep angle
μ"	=	projection of the Mach cone angle on the w-r plane
ν	=	hub-to-tip ratio; lateral sweep angle; section thickness ratio (t _{max} /c); relative thickness
Π	=	acoustic power
ρ	=	density
σ	=	sweep angle; stress; cascade solidity
τ	Ξ	shear stress; LE and TE thickness factors; time interval between successive events
τ _b	=	time for a gust to travel the swept semi- chord (b)
φ	=	local camber angle; angle between leading and trailing edge along the unwrapped conical surface
Φ	=	mean camber angle
ω	=	radian frequency; wheel rotation speed
^ω r	=	radian frequency of blade passages
Ω	=	shaft rotation frequency

SUBSCRIPTS

А	=	axial; along blade leading edge
a	=	acoustic
ax	=	axial
В	=	blade-fixed coordinates
с	=	circumferential
crit	=	critical
cg	Ξ	center of gravity
D	=	defect
G	=	gas-fixed coordinates
g	=	geometric
i	=	component parallel to blade array
i, j	=	indices of spatial coordinates
L	=	normal to leading edge; leading edge
L	=	lower
Μ	= .	mainstream
m	=	meridional component
minm	=	minimum
0	=	trace speed
R	=	radial

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r	=	radial; component normal to blade array
Т	-	tangential; wake trace
t	=	tip
u	=	upper; tangential
W	=	tangential
x	=	chordwise distance from LE; direction normal to z-axis
У	=	Cartesian coordinate normal to z-axis
Z	Ξ.	axial
ω	=	freestream relative
1,2,3	=	x, y, and z directions
1,2,3 1	E	x, y, and z directions rotor inlet station
		• • •
1	=	rotor inlet station

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