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Incorporation of Coupled-Nonequilibriu Chemistry Into a Two-Dimensional Nozzle Code (SEAGULL)

Alan W. Ratliff

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Incorporation of Coupled Nonequilibrium Chemistry Into a Two-Dimensional Nozzle Code (SEAGULL)

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SUMMARY AND INTRODUCTION

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During the past decade several excellent computer codes for analyses of complicated flow fields have been developed. NASA-Langley's SEAGULL program (Appendix A) that treats multiple shock waves and contact surfaces, using a floating shock fitting technique, is one such code. Previously, calculations performed with this code were limited to the ideal gas assumption. It is the purpose of this document to describe the extension of the code to handle real gas effects via the incorporation of a general finite rate chemistry and vibrational energy exchange package.

This document and the associated computer code (modified SEAGULL) now provide NASA-Langley with a benchmark finite rate general chemistry nozzle and plume code. The modified code retains all of its original features plus the capability to treat chemical and vibrational nonequilibrium chemistry. The chemistry package is extremely general in nature, handling any chemical reaction or vibrational energy exchange mechanism as long as thermodynamic data and rate constants are available for all participating species.

1. SYMBOLS

Symbol	Description
А	defined as used
b(z)	lower duct wall ordinate
c(z)	upper duct wall ordinate
Cp	specific heat at constant pressure
C _v	specific heat at constant volume
f(s)	entropy production (Eq.4)
f	weighting factors (Eq. 15)
F _i	species mole/mass ratio
G	defined by Eq. (50)
h	static enthalpy
Н	total enthalpy
k	reaction rate constant
К _р	equilibrium constant
P	natural log of pressure
р	pressure
q	total velocity
r	radial coordinate
R	universal gas constant
R	universal gas constant divided by global molecular weight

Symbol	Description		
S	entropy		
Т	temperature		
u	radial velocity component		
w	axial velocity component		
	net production rate of species i		
Х	transformed radial coordinate		
У	axial coordinate		
Z	axial coordinate		

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Greek

α _i	species mole fraction
β	defined by Eq.(3)
γ	ratio of specific heats
v_i', v_i''	stoichiometric coefficients
ρ	density
Ψ _i	species molecular weight
ψ	global molecular weight

Subscripts

i	refers to species i
in.	refers to initial conditions

2. DISCUSSION

2.1 BASELINE NOZZLE CODE – PROGRAM SEAGULL

Program SEAGULL is a computer code for the numerical analysis of complex two-dimensional or axisymmetric supersonic inviscid flows of a perfect gas. The fundamental limitation of this program is that the component of the Mach number in the axial direction at any point in the flow field must remain supersonic. The program was primarily designed for the analysis of internal flows. It can compute the flow field produced by a single duct or several ducts that are merged. Although designed for internal flows, it can also compute jets and plumes. The program continuously monitors the flow field to detect the formation of shock waves. All discontinuities are treated explicitly, and all interactions are treated by a locally exact solution. A description of the method used is given in Appendix A.

2.2 MODIFIED NOZZLE CODE

The modified program Seagull is now a computer code for the numerical analysis of complex two-dimensional or axisymmetric supersonic inviscid flows with fully coupled nonequilibrium chemistry. The code retains all the salient features discussed in Section 2.1, plus the additional capability to treat frozen or finite rate chemically reacting flows. The generalized chemistry package included in the modified Seagull also permits the analysis of vibrational nonequilibrium energy exchanges.

2.3 DEVELOPMENT OF EQUATIONS

The basic equations governing the flow of gases with nonequilibrium reacting chemistry are the following:

$$\nabla \cdot \rho \overline{q} = 0 \quad \text{Global Continuity}$$

$$\nabla \overline{q} \cdot \rho \overline{q} + \nabla p = 0 \quad \text{Momentum}$$

$$\rho \overline{q} \cdot \nabla H = 0 \quad \text{Energy}$$

$$\rho \overline{q} \cdot \nabla F_i - \dot{w}_i = 0 \quad \text{Species Continuity}$$

$$P = \rho \quad \Re T \sum_{i=1}^{N} \frac{\alpha_i}{\psi_i} \quad \text{Equation of State}$$

$$H = h + \frac{1}{2} \nabla (\overline{q} \cdot \overline{q}) \quad \text{Total Enthalpy}$$

$$\dot{w}_i = \frac{\rho}{\psi_i} \sum_{j=1}^{M} (\nu_{i,j}^{\prime\prime} - \nu_{i,j}^{\prime\prime}) \left[k_f \prod_{\ell=1}^{N} F_{\ell}^{\nu_{\ell}^{\prime\prime},j} - k_b \prod_{\ell=1}^{N} F_{\ell}^{\nu_{\ell}^{\prime\prime},j} \right]$$

$$\text{General Species}$$

$$\text{Production Rate}$$

$$(1)$$

where N species are involved in M reactions.

These equations are expanded in a Cartesian coordinate system, scaled (i.e., nondimensionalized) with respect to some initial reference conditions, and are then solved in the modified SEAGULL code.

To avoid unnecessarily complicating the discussion, the remainder of the derivation is shown for a two-dimensional case only. The code, however, can also handle axisymmetrics flows. Scaled and expanded, the equations then become

$$\beta P_{z} + wu_{r} + A_{1} wu P_{r} - uw_{r} - \frac{w}{A_{2}} f(s) = 0$$

$$\beta w_{z} + \frac{T}{\psi} u_{r} + A_{1} wu w_{r} - A_{1} \frac{T}{\psi} u P_{r} + \frac{T}{A_{2}\psi} f(s) = 0$$

$$u_{z} + \frac{u}{w} u_{r} + \frac{T}{w\psi} P_{r} = 0$$

$$wF_{i_{z}} + uF_{i_{r}} - \frac{\dot{w}}{\rho} = 0$$

$$H_{z} + \frac{u}{w} H_{r} = 0$$
(2)

where the subscripts refer to differentiation with respect to the r, z coordinates.

 β , A_1 and A_2 are conveniently defined as follows:

$$\beta = A_1 w^2 - \frac{T}{\psi}$$

$$A_1 = 1 - \frac{\gamma_{\text{in}} - 1}{A_2}$$

$$A_2 = \gamma_{\text{in}} C_p$$

$$(3)$$

The function f(s) is the entropy production due to finite rate chemical reactions.

$$f(s) = (q \cdot \nabla s)_{chem}$$
(4)

The general set of governing equations (Eq. (2)) are then recast in terms of the computational coordinates (X, Z) that are defined by

$$X = (r - b(z))/(c(z) - b(z))$$
(5)

Z = Z

and

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where b(z) and c(z) define the lower and upper walls of the duct or nozzle.

All the primary variables in the equation set are scaled with respect to the initial conditions. The pressure, density, temperature and molecular weight are non-dimensionalized by p_{in} , ρ_{in} , T_{in} , ψ_{in} , respectively. Velocity is conveniently scaled by $\sqrt{p_{in}/\rho_{in}}$ and entropy is non-dimensionalized by the specific heat at constant volume $C_{v_{in}}$, i.e., $S = ([S] - [S_{in}])/C_{v_{in}}$.

The specific heat at constant pressure, C_p , is scaled with respect to its initial value, $[C_p] = [C_p] / [C_{p_{in}}]$. Static enthalpy is non-dimensionalized by $C_{p_{in}}$ and for consistency the total enthalpy equation then becomes

$$H_{T} = h + \left[\frac{\left[\mathcal{R}\right]}{\left[C_{p_{in}}\right]} \left(u^{2} + w^{2}\right)\psi$$
(6)

where all values are non-dimensional except $[\Re]$ and $[C_{p_{in}}]$.

The finite rate chemistry parameters, of course, have dimensional characteristics. Interrelationships between the flow field and the finite rate reactions are treated internally by temporarily dimensioning the variables appearing in the species continuity equations. The reacting chemistry calculation is then carried out and the results are subsequently non-dimensionalized.

2.4 SOLUTION TECHNIQUE

A finite difference numerical scheme utilizing the explicit MacCormack operator is used to integrate the equations for conservation of global mass, momentum and energy. The species conservation equations are solved by an implicit technique similar to that developed by Moretti (Ref. 1). This method was chosen to provide good stability characteristics in the overall numerical scheme.

The general solution technique is discussed adequately in Appendix A. The same methodology is retained in the modified SEAGULL with the finite rate reacting chemistry. The only exception is the species continuity equation set which is solved by the following technique. A detailed description of the rate processes that occur in finite rate reacting flows requires that a myriad of mechanisms be considered to include all the possible chemical and vibrational reactions of dissociation, formation, recombination, etc.

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> All of these, however, can be treated with a very general formalism. In the form usually quoted in chemical kinetics (Ref. 2) the phenomenological law of mass action states that the rate of a reaction is proportional to the product of the concentrations of the reactants. Thus, for a general reaction of the form

$$\sum_{j=1}^{N} \nu_j A_j \Longrightarrow \sum_{j=1}^{N} \nu_j^{"} A_j$$
(7)

the net rate of production \dot{w}_i for any participating species for which the stoichiometric coefficients ν'_i and ν''_i are not equal can then be written as

$$\dot{w}_{i} = \frac{\rho}{\Psi_{i}} \sum_{j=1}^{M} (\nu''_{i,j} - \nu'_{i,j}) \begin{bmatrix} N & \nu'_{\ell,j} & N \\ R & \Pi & F_{\ell} & k_{b} & \prod_{\ell=1}^{N} F_{\ell} & j \end{bmatrix}$$
(8)

Assuming small deviations from equilibrium, the forward and backward reaction rate constants, k_f and k_b , respectively, can be related to the concentration equilibrium constant and to the pressure equilibrium constant as follows:

$$\frac{k_{f}}{k_{b}} = K_{c} = K_{p} (\Re T)^{i=1} (\nu_{i} - \nu_{i}'')$$
(9)

The significance of the pressure equilibrium constant K_p is that it can be easily evaluated for any reaction using tabulated values of K_f the equilibrium constant for formation from the elements. Values of K_f are commonly tabulated in conjunction with specific heats, entropies and enthalpies as a function of temperature, and are available in general for most species. An equally convenient method exists for determining K_p from the change of free energy accompanying the reaction, i.e.,

$$K_{p} = \exp\left(-\Delta G/\Re T\right)$$
(10)

where ΔG is the change in free energy during the reaction process. Free energy values are also available for most species in tabular form. This method is used to compute K_p in Program SEAGULL. The JANNAF thermochemical tables (Ref.4) are used as the source for obtaining input data for various chemical systems. ΔG is computed directly by taking the difference in free energy between the products and reactants.

For reasons of computational speed and efficiency, the program contains explicit expressions, as obtained from Eq. (11), for the most commonly encountered reaction mechanisms. Twelve types of reaction mechanisms are considered as possible contributors to the calculation of the net rate of production, \dot{w}_i .

Reaction Type				
(1,7)	A + B	<u></u>	C + D	
(2,8)	A + B + M	<u> </u>	C + M	
(3, 9)	A + B	<u> </u>	C + D + E	
(4,10)	A + B		С	(11)
(5,11)	A + M		C + D + M	
(6,12)	A + M	1	C + M	

Reaction types (7) through (12) correspond to reaction types (1) through (6), but proceed in the forward direction only.

To reduce roundoff and truncation errors, the forward and backward rates for each reaction are computed separately. All contributions to the molar rate of production of a given species are then computed and added algebraically to form matrix coefficients (discussed later). Since reaction types (7) through (14) proceed in the forward direction only, the second term on the right-hand side of Eq. (11) is disregarded in calculating the contributions to the coefficient matrix.

In reactions (2), (5) and (6) as well as in (8), (11) and (12), M denotes a third body species which can be specified. For these reactions the situation often occurs where for various third bodies the respective rate constants differ only by a constant multiplier. These multipliers can be considered as third body efficiencies or weighting factors. If such a case is encountered, the third body species mole mass ratio $F_{\rm M}$ becomes effectively a fictitious mole mass ratio, consisting of the weighted sum over all those species having a nonzero weighting factor, i.e.,

$$\mathbf{F}_{\mathbf{M}} = \sum_{\mathbf{i}} \mathbf{f}_{\mathbf{i}} \mathbf{F}_{\mathbf{M}_{\mathbf{i}}}$$
(12)

where f; are the weighting factors.

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The forward rate constant, k_f , is generally a function of temperature. It is most often expressed in Arrhenius form. Again, for speed and efficiency in computation, the rate constants are divided into five types:

Rate Constant Type

(1)
$$k_{f} = A$$

(2) $k_{f} = AT^{-N}$
(3) $k_{f} = A \exp(B / \Re T)$ (13)
(4) $k_{f} = AT^{-N} \exp(B / \Re T)$
(5) $k_{f} = AT^{-N} \exp(B / \Re T^{M})$

The equations presented in this document provide a very general formalism for the evaluation of various rate processes. The specification of particular systems and associated rate constant will be up to the program user.

Consider now the general species continuity equation

$$\rho \overline{q} \cdot \nabla F_{i} = \dot{w}_{i}$$
(14)

and making use of the foregoing discussion of the rate process we now proceed to describe a calculational technique for determining the individual species composition on a point-by-point basis. The description of this process is substantially simplified if Eq. (11) is specialized to a particular reaction type, say number (7) from Eq. (14) which is a one-way, two-body reaction.

$$A + B \longrightarrow C + D \tag{15}$$

the net production rate for this process is

$$\dot{w} = -k_f \rho^2 F_A F_B$$
(16)

and the species continuity equation for species B then becomes

$$\rho \overline{q} \cdot \nabla F_{B} = -k_{f} \rho^{2} F_{A} F_{B}$$
 (17)

This equation can readily be solved using finite difference techniques employing explicit relationships, such as Euler or more sophisticated schemes, such as Runge-Kutta. The step size for integrating this equation, however, is severely limited by stability criteria. It can be seen from Eq. (20) that the rate of change of a species along the streamline becomes increasingly larger as the flow speed is slowed, the density increased, or for fast reaction rates. In rocket engine problems, combinations of slow speeds, high densities and fast reaction rates (i.e., quasi-equilibrium) are quite common and integration step sizes so small (i.e., < 10⁻⁸ meters) are encountered that the solution becomes impractical in terms of computation time. For this reason, the technique described in Ref.3 based on a linearization of the production rates was utilized. Writing Eq. (20) in finite difference form over an integration step from station n at $z = z_0$ to n+l at $z = z_0 + \Delta z$

$$F_{B_{n+1}} = F_{B_n} - \frac{k_f \Delta z \rho}{q} F_{A_{n+1}} F_{B_{n+1}} - \frac{u}{w} \Delta z \frac{\partial F_B}{\partial r}$$
(18)

And evaluating all the species concentrations at the downstream point results in a set of simultaneous nonlinear algebraic equations. The lateral gradient term $\frac{\partial F_B}{\partial r_n}$ is evaluated at the upstream station n and uses windward differences. In order to solve these equations we must then linearize the term $F_{A_{n+1}}F_{B_{n+1}}$ which is accomplished following the lead of Ref. 3. If this term is expanded in terms of its values at station n along with the increments over n to n+1 we can obtain the following expression.

$$F_{A_{n+1}}F_{B_{n+1}} = F_{A_n}F_{B_{n+1}} + F_{B_n}F_{A_{n+1}} - F_{A_n}F_{B_n}$$
 (19)

neglecting products of differentials which are assumed to be of second order importance. Equation (21) can now be written in its linearized form. Let $C = \Delta z \ k_f \rho/w$, then

$$\mathbf{F}_{\mathbf{B}_{n+1}} = \mathbf{F}_{\mathbf{B}_n} - \mathbf{C} \quad \left[\mathbf{F}_{\mathbf{A}_n} \mathbf{F}_{\mathbf{B}_{n+1}} + \mathbf{F}_{\mathbf{B}_n} \mathbf{F}_{\mathbf{A}_{n+1}} - \mathbf{F}_{\mathbf{A}_n} \mathbf{F}_{\mathbf{B}_n} \right] - \frac{\mathbf{u}}{\mathbf{w}} \Delta z \frac{\partial \mathbf{F}_{\mathbf{B}}}{\partial \mathbf{r}_n}$$

and similarly,

$$\mathbf{F}_{\mathbf{A}_{n+1}} = \mathbf{F}_{\mathbf{A}_n} - \mathbf{C} \quad \left[\mathbf{F}_{\mathbf{A}_{n+1}} \mathbf{F}_{\mathbf{B}_n} + \mathbf{F}_{\mathbf{B}_{n+1}} \mathbf{F}_{\mathbf{A}_n} - \mathbf{F}_{\mathbf{A}_n} \mathbf{F}_{\mathbf{B}_n} \right] - \frac{\mathbf{u}}{\mathbf{w}} \Delta \mathbf{z} \frac{\partial \mathbf{F}_{\mathbf{A}}}{\partial \mathbf{r}_n}$$

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(20)

Equation (20) can then be expressed in terms of a set of unknowns and calculable coefficients, C. Rewriting these we obtain

$$F_{B_{n+1}} = Q_{B_{n}} - C F_{A_{n}} (F_{B_{n+1}}) - C F_{B_{n}} (F_{A_{n+1}})$$

$$F_{A_{n+1}} = Q_{A_{n}} - C F_{B_{n}} (F_{A_{n+1}}) - C F_{A_{n}} (F_{B_{n+1}})$$
(21)

......

where

$$Q_{i_{n}} = F_{i_{n}} + CF_{i_{n}}F_{j_{n}} - \frac{u}{w}\Delta z \frac{\partial F_{i}}{\partial r_{n}}$$

$$F_{A_{n+1}} (1 + CF_{B_{n}}) + (CF_{A_{n}})F_{B_{n+1}} = Q_{A_{n}}$$

$$F_{A_{n+1}} (CF_{B_{n}}) + (1 + CF_{A_{n}})F_{B_{n+1}} = Q_{B_{n}}$$
(22)
(23)

~ ---

A matrix can now be formed using totally known information.

$$\begin{bmatrix} 1 + C F_{B_{n}} & C F_{A_{n}} \\ C F_{B_{n}} & 1 + C F_{A_{n}} \end{bmatrix} \begin{bmatrix} F_{A_{n+1}} \\ F_{B_{n+1}} \end{bmatrix} = \begin{bmatrix} Q_{A_{n}} \\ Q_{B_{n}} \end{bmatrix}$$
(24)

The matrix [A] [X] = [B] is then solved for the unknown compositions $F_{A_{n+1}}$, $F_{B_{n+1}}$ via a triangulation technique. Although consuming more time per n+1 integration step than an explicit formulation, the implicit technique employed here is unconditionally stable permitting much larger step sizes, thus allowing solutions to be obtained for problems where the small steps required by the explicit technique prevented even the consideration of the case. Finally it should be recalled that an extremely simple case was chosen only for purposes of illustration and the general technique coded in the modified Program SEAGULL will handle many species with multiple reactions.

2.5 BOUNDARY CONDITIONS

By combining the two momentum equations we obtain

$$\boldsymbol{\tau}_{z} + w^{2} \frac{\boldsymbol{\tau}A_{1}}{\beta} \boldsymbol{\tau}_{r} + \left(\frac{A_{1} w^{2} \boldsymbol{\tau}^{2}}{\beta} + 1\right) \frac{T}{\psi w^{2}} P_{r} - \frac{T}{\beta \psi} \frac{\boldsymbol{\tau}}{wA_{2}} f(s) = 0 \quad (25)$$

where $\boldsymbol{\tau} = u/w$.

The global continuity equation is used in the form

$$P_{z} + \frac{w^{2}}{\beta} (\tau A_{1}P_{r} + \tau_{r}) - \frac{w}{A_{2}\beta} f(s) = 0.$$
 (26)

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Now, using Eqs. (28) and (29) that involve partials of P and τ only, a characteristics compatibility equation can be written

$$\frac{d\tau}{dz} \neq \frac{T}{w^2 \psi} \sqrt{\frac{A_1(u^2 + w^2)}{T/\psi} - 1} \frac{dP}{dz} = RHS$$
(27)

where the right-hand side (RHS) is:

RHS =
$$\frac{f(s)}{A_2} \left[\frac{T}{\beta \psi w} \left(\tau \pm \sqrt{\frac{A_1(u^2 + w^2)}{T/\psi} - 1} \right) \right]$$
 (28)

Then by combining these equations with the chain rule definitions,

$$\frac{\mathrm{dP}}{\mathrm{dz}} = \frac{\partial P}{\partial z} + \lambda \frac{\partial P}{\partial r}$$

$$\frac{\mathrm{d\tau}}{\mathrm{dz}} = \frac{\partial \tau}{\partial z} + \lambda \frac{\partial \tau}{\partial r}$$
(29)

an equation for the axial pressure variation is obtained in terms of known quantities.

$$P_{z} = -\lambda P_{r} + \frac{w^{2} (RHS - \tau_{z} - \lambda \tau_{r})}{\frac{T}{\psi} \sqrt{\frac{A_{1}(u^{2} + w^{2})}{T/\psi} - 1}}$$
(30)

where

$$\lambda = \frac{w^2 \tau A_1}{\beta} + \frac{T}{\beta \psi} \sqrt{\frac{A_1 (u^2 + w^2)}{T/\psi} - 1}$$
(31)

By integrating Eq. (30), the pressure is obtained at the new wall point and the velocity is computed from the streamwise momentum equation

$$\rho q \frac{\partial q}{\partial s} = - \frac{\partial P}{\partial s}$$
(32)

Velocity components u and w are then obtained from the known wall slope. The boundary conditions are completed by integrating the species continuity equations along the wall in a manner similar to that used for the interior points.

2.6 SHOCK WAVES

The general scheme outlined in Appendix A is utilized for computing shock wave information. The original ideal gas analysis was supplemented by the addition of methodology to handle reacting gas calculations across the shocks.

Figure 1 illustrates a stream tube passing through an oblique shock wave. This wave, which is extremely thin, will cause an almost instantaneous rise in pressure and temperature. For some distance downstream of the shock wave (in a reacting gas) a non-equilibrium zone will exist followed by a return to chemical equilibrium. The following analysis discusses the fluid flow properties in such a way that the non-equilibrium process need not be specified in order to arrive at an exact solution for the gas properties. It is impossible to determine the location of the new equilibrium shock point location without a detailed description of the non-equilibrium reaction process. It will be assumed therefore that this zone is thin and that no significant errors are introduced by letting the downstream physical location lie on the upstream location.

Consider a control surface as shown in the figure. The conservation of mass yields:

$$\rho_2 q_2 A_2 - \rho_1 q_1 A_1 = 0 \tag{33}$$

Let ns and ts be unit vectors normal and tangent to the shock surface, the unit vectors parallel and perpendicular to the streamline upstream of the shock are

and the unit vectors parallel and perpendicular to the streamline downstream of the shock are:

$$\overline{\ell \, \theta'} = \sin \left(\epsilon - \delta \right) \, \overline{\mathrm{ns}} \, + \cos \left(\epsilon - \delta \right) \, \overline{\mathrm{ts}}$$

$$\overline{l n'} = -\cos(\epsilon - \delta) \overline{ns} + \sin(\epsilon - \delta) \overline{ts}$$
(35)

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Conservation of momentum gives

$$-(\mathbf{p}_{1}+\rho_{1}\mathbf{q}_{1}^{2})\mathbf{A}_{1}\overline{\boldsymbol{\ell}\boldsymbol{\theta}}+\frac{\mathbf{p}_{1}\mathbf{A}_{1}}{\tan\boldsymbol{\epsilon}}\overline{\boldsymbol{\ell}\mathbf{n}}+(\mathbf{p}_{2}+\rho_{2}\mathbf{q}_{2}^{2})\mathbf{A}_{2}\overline{\boldsymbol{\ell}\boldsymbol{\theta}^{\dagger}}-\frac{\mathbf{p}_{2}\mathbf{A}_{2}\overline{\boldsymbol{\ell}\mathbf{n}}}{\tan(\boldsymbol{\epsilon}-\boldsymbol{\delta})}$$
(36)

and, after substitution of Eqs. (34) and (35) and setting each component to zero, Eq. (36) becomes,

$$\rho_2 q_2^2 \cos(\epsilon - \delta) A_2 - \rho_1 q_1^2 A_1 \cos\epsilon = 0$$

$$(p_1 + \rho_1 q_1^2) A_1 \sin\epsilon + \frac{p_1 A_1}{\tan\epsilon} \cos - (p_2 + \rho_2 q_2^2) A_2 \sin(\epsilon - \delta)$$

$$p_1 A_2 \cos(\epsilon - \delta)$$

$$(37)$$

$$-\frac{P_2 A_2 \cos{(\epsilon - \delta)}}{\tan{(\epsilon - \theta)}} = 0$$
(38)

But from geometry it can be seen that

$$\frac{A_2}{A_1} = \frac{\sin(\epsilon - \delta)}{\sin\epsilon}$$
(39)

After substitution of Eqs. (40); (33), (37) and (38) become

$$\rho_2 q_2 \sin(\epsilon - \delta) - \rho_1 q_1 \sin\epsilon = 0 \tag{40}$$

$$\rho_2 q_2^2 \sin(\epsilon - \delta) \cos(\epsilon - \delta) - \rho_1 q_1^2 \sin\epsilon \cos\epsilon = 0$$
(41)

$$p_{2} + \rho_{2} q_{2}^{2} \sin^{2}(\epsilon - \delta) - p_{1} - \rho_{1} q_{1}^{2} \sin^{2} \epsilon = 0$$
 (42)

The above set of relations contains ϵ , δ , p_2 , ρ_2 , q_2 as unknown quantities, but

$$p_2 = p(s_2, q_2) ; \rho_2 = \rho(s_2, q_2)$$
 (43)

So that if one variable, say ϵ , is taken as an independent parameter the remaining unknowns (δ, q_2, s_2) may be found by an iterative solution. These equations are, of course, formally the same as the ideal gas solution. The difference lies only in the variation of pressure etc., with entropy and velocity.

2.6.1 Iterative Solution of the Oblique Shock Relations

Rearranging Eq. (44) yields;

$$\sin(\epsilon - \delta) = \frac{\rho_1 q_1 \sin\epsilon}{\rho_2 q_2}$$

while squaring both sides of Eq. (45) and substituting the above relations yields, after simplification;

$$q_2 - q_1 \left\{ \left(\frac{\rho_1}{\rho_2} \right)^2 \sin^2 \epsilon + \cos^2 \epsilon \right\}^{1/2} = 0$$
(44)

and Eq. (42) becomes

$$p_{2} + \rho_{1} q_{1}^{2} \sin \epsilon \left\{ \frac{\rho_{1}}{\rho_{2}} - 1 \right\} - p_{1} = 0$$
(45)

In functional form Eqs. (44) and (45) are just

$$G_1(s_2, q_2) = 0$$

 $G_2(s_2, q_2) = 0$ (46)

(48)

From calculus

$$dG_{1} = \frac{\partial G_{1}}{\partial q_{2}} dq_{2} + \frac{\partial G_{1}}{\partial s_{2}} ds_{2}$$
$$dG_{2} = \frac{\partial G_{2}}{\partial q_{2}} dq_{2} + \frac{\partial G_{2}}{\partial s_{2}} ds_{2}$$
(47)

Now

$$\frac{\partial G_{1}}{\partial q_{2}} = 1 + q_{1} \left(\frac{q_{1}}{q_{2}}\right) \left(\frac{\rho_{1}}{\rho_{2}}\right)^{2} \sin^{2} \epsilon \frac{\partial}{\partial q_{2}} \left(\ell n \rho_{2}\right)$$

$$\frac{\partial \mathbf{G}_{1}}{\partial \mathbf{s}_{2}} = \mathbf{q}_{1} \left(\frac{\mathbf{q}_{1}}{\mathbf{q}_{2}} \right) \left(\frac{\rho_{1}}{\rho_{2}} \right)^{2} \quad \sin^{2} \boldsymbol{\epsilon} \quad \frac{\partial}{\partial \mathbf{s}_{2}} \left(\ell_{\mathrm{I}} \rho_{2} \right)$$

$$\frac{\partial G_2}{\partial q_2} = p_2 \frac{\partial}{\partial q_2} \left(ln p_2 \right) - \left(\frac{\rho_1}{\rho_2} \right)^2 q_1^2 \sin^2 \epsilon \rho_2 \frac{\partial}{\partial q_2} \left(ln \rho_2 \right)$$

$$\frac{\partial G_2}{\partial s_2} = p_2 \frac{\partial}{\partial s_2} \left(\ln p_2 \right) - \left(\frac{\rho_1}{\rho_2} \right)^2 \sin^2 \epsilon \rho_2 \frac{\partial}{\partial s_2} \left(\ln \rho_2 \right)$$

Rather than calculate the partial derivatives numerically by perturbing the functions $\ln p_2$, $\ln \rho_2$ approximate values for these derivaties will be found by assuming that locally the gas behaves ideally, that is to say

$$\frac{\partial R_2}{\partial s_2} = \frac{\partial R_2}{\partial q_2} = \frac{\partial \gamma_2}{\partial s_2} = \frac{\partial \gamma_2}{\partial q_2} = 0$$

so that holding q_2 constant

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$$\frac{\partial(\ln p_2)}{\partial s_2} \cong \frac{\partial(\ln \rho_2)}{\partial s_2} \cong \frac{1}{1 - \gamma_{\text{in}}}$$
(49)

and holding s_2 constant

$$\frac{\partial(\ln p_2)}{\partial q_2} \cong \gamma_2 \frac{\partial}{\partial q_2} (\ln \rho_2) \cong -\frac{q_2 \rho_2}{p_2}$$
(50)

writing Eq. (47) in finite difference form:*

$$G_{1}^{(n+1)} - G_{1}^{(n)} = \frac{\partial G_{1}^{(n)}}{\partial q_{2}} \left(q_{2}^{(n+1)} - q_{2}^{(n)} \right) + \frac{\partial G_{1}^{(n)}}{\partial s_{2}} \left(s_{2}^{(n+1)} - s_{2}^{(n)} \right)$$
$$G_{2}^{(n+1)} - G_{2}^{(n)} = \frac{\partial G_{2}^{(n)}}{\partial q_{2}} \left(q_{2}^{(n+1)} - q_{2}^{(n)} \right) + \frac{\partial G_{2}^{(n)}}{\partial s_{2}} \left(s_{2}^{(n+1)} - s_{2}^{(n)} \right)$$

Since the root $G_1 = G_2 = 0$ is desired, $G_1^{(n+1)}$, $G_2^{(n+1)}$ are set to zero, resulting in

where n here is an iteration counter.

$$s^{(n+1)} = s^{(n)} + \left(G_2^{(n)} \frac{\partial G_1^{(n)}}{\partial q_2} - G_1^{(n)} \frac{\partial G_2^{(n)}}{\partial q_2}\right)$$

$$\left(\frac{\partial G_1^{(n)}}{\partial s_2} \frac{\partial G_2^{(n)}}{\partial q_2} - \frac{\partial G_2^{(n)}}{\partial s_2} \frac{\partial G_1^{(n)}}{\partial q_2}\right)$$
(51)

and

$$q_{2}^{(n+1)} = q_{2}^{(n)} - \left\{ G_{1}^{(n)} + \frac{\partial G_{1}^{(n)}}{\partial s_{2}} \left(s^{(n+1)} - s^{(n)} \right) \right\} / \frac{\partial G_{1}^{(n)}}{\partial q_{2}}$$
(52)

The iterative solution using Eqs.(51) and (52) is continued until the desired convergence of G_1 and G_2 is reached. The solution is completed by

$$\delta = \epsilon - \sin^{-1} \left\{ \frac{\rho_1 q_1}{\rho_2 q_2} \sin \epsilon \right\}$$
(53)

The first guess to start the solution is an ideal gas solution to the set of equations. If it is indeed an ideal gas under analysis the first guess is exact. These relations are

$$\delta = \epsilon - \tan^{-1} \left\{ \tan \left(\frac{1}{M_1^2 \sin^2 \epsilon} + \frac{\gamma_1 - 1}{2} \right) \frac{2}{\gamma_1 + 1} \right\}$$

$$q_2 = q_1 \frac{\cos\epsilon}{\cos(\epsilon - \delta)}$$
(54)

$$\mathbf{s}_{2} = \mathbf{s}_{1} + \frac{\mathbf{R}_{1}}{\gamma_{1}-1} \left\{ \ln \left[\frac{2\gamma_{1}\mathbf{M}_{1}^{2} \sin^{2} \boldsymbol{\epsilon} - (\gamma_{1}-1)}{\gamma_{1}+1} \right] + \gamma_{1} \ln \left[\frac{\tan(\boldsymbol{\epsilon}-\boldsymbol{\delta})}{\tan \boldsymbol{\epsilon}} \right] \right\}$$

2.7 CONTACT SURFACES

Ref and

Contact surfaces or slip lines are computed utilizing the condition that the pressures and flow angles must match across the surface. The equations for pressure and flow angle developed for the wall boundary conditions are also applied at the contact surface except that the streamline slope must now be obtained iteratively utilizing the matching conditions. Equations (25) and (26) are transformed to the computational coordinates and integrated via the MacCormack operator as described in Appendix A. The predictor (first level) is applied to the high Mach number side of the slip line and the corrector (second level) is applied to the low Mach number side.

With the pressure and slope now known at the contact surface the velocity is again obtained from Eq. (35). The velocity components are then evaluated from the known slip line slope.

2.8 THERMODYNAMIC PROPERTIES

The modified Program SEAGULL handles general gases composed of various chemical species rather than a global gas. This requires thermodynamic data inputs for all participating species as well as for all vibrational levels of the various excited molecules involved in vibrational energy exchange. For these molecules it is assumed that the molecular rotation remains in equilibrium with the translational temperature but each molecule is allowed to vibrate independently, corresponding to the energy in its respective vibrational level.

Thermodynamic data are supplied to the code in the form of tabular inputs of specific heat, entropy and enthalpy for each species as a function of temperature and are taken directly from the JANAF thermochemical tables (Ref. 4).

2.9 VIBRATIONAL NONEQUILIBRIUM

To make maximum use of the current technology of finite rate formulations, the various vibrational levels of the respective excited molecules are treated as individual chemical species. In order to treat them as individual species, it is first assumed that the excited molecule can be resolved into its various vibrational levels each of which is then treated as a separate species. This assumption requires that the thermodynamic properties be separately specified for each vibrational level. It is further assumed that molecular rotation remains in equilibrium with the translational temperature but the molecules can vibrate independently of the translational temperature. Accordingly, each molecule is permitted to vibrate independently corresponding to the energy in its respective vibrational level.

Appendix A

SHOCK FITTING METHOD FOR COMPLICATED TWO-DIMENSIONAL SUPERSONIC FLOWS

Shock Fitting Method for Complicated Two-Dimensional Supersonic Flows

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The floating shock fitting technique is examined. Second-order difference formulas are developed for the computation of discontinuities. A procedure is developed to compute mesh points that are crossed by discontinuities. The technique is applied to the calculation of internal two-dimensional flows with arbitrary number of shock waves and contact surfaces. A new procedure, based on the coalescence of characteristics, is developed to detect the formation of shock waves. Results are presented to validate and demonstrate the versatility of the technique.

I. Introduction

INVISCID supersonic flows are governed by a first-order quasi-linear hyperbolic system of equations. The numerical computation of these flows is complicated, because the regular solution may break down due to the nonlinearity of the governing equations. The breakdown is characterized by the appearance of surfaces, such as shock waves and vortex sheets, across which the dependent variables or their derivatives are discontinuous. During the last decade, 2 numerical techniques have emerged for the analysis of these flows. One, known as shock capturing, tries to remove the explicit computation of the discontinuties by generalizing the concept of a solution of the Euler equations to include weak solutions (i.e., discontinuities). Because the shock capturing scheme requires no special treatment to deal with discontinuties, it has become an extremely popular way of computing. However, despite its present popularity, the results obtained with this technique¹ force us to agree with Moretti's conclusion that shock capturing is a poor interpretation of a physical phenomenon, and an extremely uneconomical way of computing.3

The other technique, known as shock fitting, makes special provisions for explicitly computing the discontinuties. In essence, it locates the discontinuties and treats them as boundaries between regions where a regular solution is valid. The effectiveness and soundness of this approach has been proven for a large number of problems involving 2, 3, and 4 independent variables. However, for problems in 3 or more independent variables, the partitioning of a flowfield into regions where a regular solution is valid, can create some difficult topological problems.³ Recently, Moretti⁴ has developed a technique (reminiscent of that proposed by Richtmyer and Morton,⁵ that treats the discontinuities explicitly but does not require them as boundaries of the flow. This therefore eliminates the problems associated with partitioning the flowfield. It is this technique, known as floating shock fitting, that forms the subject matter of this paper.

To avoid unnecessary complications, the technique will be discussed in the context of a two-dimensional problem. However, to demonstrate the capabilities of the technique, results from complex flowfields are presented.

II. Problem Definition

We will discuss the floating shock fitting technique as it applies to the flow of an inviscid, perfect gas through a twodimensional duct of arbitrary geometry. The flow is assumed to be supersonic throughout, and consequently, irregular shock reflections are not considered. Shock waves generated by either discontinuties in the slopes of the walls, or by coalescence of pressure waves, will be automatically detected and fitted into the calculation. The number of shock waves, contact surfaces, and the number of interactions of these discontinuties, is considered arbitrary (limited only by computer storage).

III. Governing Equations

The flow variables are nondimensionalized by scaling the pressure, density, and temperature, with respect to their initial values (p_i, p_i, T_i) , scaling the velocities with respect to $(p, /p_i)^{1/2}$, and scaling all lengths with respect to some arbitrary reference length. The Euler equations written in a Cartesian frame (r, z), with the z-axis running along the length of the duct, are

$$P_{z} + (uwP_{r} + \gamma wu_{r} - \gamma uw_{r})/\beta = 0$$

$$w_{z} + (uww_{r} - \gamma Tu_{r} - TuP_{r})/\beta = 0$$

$$u_{z} + (uu_{r} + TP_{r})/w = 0$$

$$S_{z} + uS_{z}/w = 0$$
(1)

Where P is the natural logarithm of pressure, u and w are the velocity components in the r and z directions, respectively; S is the entropy, γ is the ratio of specific heats; the temperature is given by

$$T = \exp((S + (\gamma - 1)P)/\gamma)$$
(2)

and $\beta = w^2 - \gamma T$.

The following equations are introduced here for later reference. By combining the 2 momentum equations, we get

$$\tau_z + \frac{w^2 \tau \tau_r}{\beta} + \left(I + \frac{\tau^2 w^2}{\beta}\right) \frac{T P_r}{w^2} = 0$$
(3a)

where $\tau = u / w$. With Eq. (3a) and the continuity equation in the form

$$P_z + w^2 (\tau P_r + \gamma \tau_r) / \beta = 0 \tag{3b}$$

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we can obtain the compatibility equation

$$\tau' \pm ([\gamma T(\beta + w^2 \tau^2)]^{\frac{1}{2}} / \gamma w^2) P' = 0$$
(4)

where the primes denote differentiation along the characteristic directions

$$\lambda^{\star} = (w^2 \tau \pm [\gamma T(\beta + w^2 \tau^2)]^{\nu_1})/\beta$$
(5)

The Euler Eqs. (1) are recast in terms of computational coordinates (X, Z) defined by

$$X = (r - b(z)) / (c(z) - b(z)) \quad Z = z$$
(6)

where r = b(z) and r = c(z) define the lower and upper walls of the duct. In matrix form the equations of motion become

$$f_{\mathsf{Z}} = \underline{A} f_{\mathsf{X}} \tag{7}$$

where the vector f is

$$f = \begin{bmatrix} P \\ u \\ w \\ S \end{bmatrix}$$
(8)

and the matrix A is

$$\underline{A} = \begin{bmatrix} a_{11} & a_{12} & a_{13} & 0 \\ a_{21} & a_{22} & 0 & 0 \\ a_{31} & a_{32} & a_{11} & 0 \\ 0 & 0 & 0 & a_{22} \end{bmatrix}$$
(9)

$$a_{11} = -(uwX_r + \beta X_z)/\beta$$

$$a_{12} = -\gamma wX_r/\beta$$

$$a_{13} = \gamma uX_r/\beta$$

$$a_{21} = -TX_r/w$$

$$a_{22} = -(uX_r + wX_z)/w$$

$$a_{31} = uTX_r/\beta$$

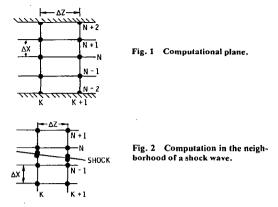
$$a_{32} = \gamma TX_r/\beta$$

IV. Interior Point Computation

Unlike shock fitting, floating shock fitting does not subdivide the computational plane into regions bounded by discontinuities; instead, a single region is used, as illustrated in Fig. 1. In this single region, mesh points are equally spaced, and all discontinuities are allowed to move freely or float over the fixed grid. Their motion is subject only to the governing equations.

Except for the mesh points at the walls and in the neighborhood of a discontinuity, all mesh points are computed using the MacCormack scheme⁶ to integrate Eq. (7). For example, consider the section of a duct, in the computational plane, sketched in Fig. 1. Using the MacCormack scheme, and assuming the solution is known at the Kth steps, $f_{N,K+1}$ is obtained from

$$f_{N,K+1} = f_{N,K} + A_{N,K} (f_{N+1,K} - f_{N,K}) \Delta Z / \Delta X$$



level 2

$$f_{N,K+1} = \frac{1}{2} \left(f_{N,K} + \tilde{f}_{N,K+1} + \tilde{A}_{N,K+1} \right)$$

$$(f_{N,K+1} - \tilde{f}_{N-1,K+1}) \Delta Z / \Delta X$$
(10)

Because the discontinuties are allowed to float over the fixed grid, special provisions must be taken when evaluating the X derivatives of f, at mesh points in the neighborhood of a discontinuity. Consider again the computation of point (N, K+1), but now in the presence of a shock as shown in Fig. 2. There is no difficulty in computing the first level of the MacCormack scheme, since the forward difference

$$f_X = (f_{N+I,K} - f_{N,K}) / \Delta X$$

is valid. However, for the second level, the backward difference

$$\tilde{f}_X = (\tilde{f}_{N,K+1} - \tilde{f}_{N-1,K+1}) / \Delta X$$

is not allowed because f is not differentiable across the shock. On the other hand, the very simple approximation,

$$\tilde{f}_{X} = (\tilde{f}_{N,K+1} - \tilde{f}_{s,K+1}) / (X_{N} - \tilde{X}_{s,K+1})$$

where s denotes conditions at the shock, can result in large truncation errors, and the computation becomes unstable. Moretti recommends using an approximation that involves the values at the shock and the 3 neighboring mesh points [Eq. (61) in Ref. 2]. We have not used this approximation, because it makes the numerical domain of dependence point (N, K+1) much larger than it physically is. Instead, we use the following approximation

$$\vec{f}_{X} = -\left[(0.5\epsilon - 1)\vec{f}_{N+1,K+1} + 0.5\vec{f}_{5,K+1} + 0.5(1 - \epsilon)\vec{f}_{N,K+1} \right] / \Delta X$$
(11)

where

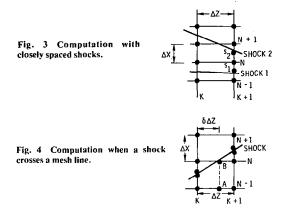
$$\boldsymbol{\epsilon} = (\boldsymbol{X}_{N,K+1} - \bar{\boldsymbol{X}}_{s,K+1}) \, / \Delta \boldsymbol{X}$$

The truncation error for Eq. (11), (based on f_X , not f) is given by

$$(\epsilon+2)(\epsilon-1)\Delta X/4f_{XX}$$

With Eq. (11), the computation of point (N,K+1) proceeds as the computation of any other interior point. Similar considerations apply to the computation of point (N-1,K+1).

If point (N+1, K+1) is not available when applying Eq. (11), as is the case indicated in Fig. 3 where 2 shocks are



shown close to each other, then if the distance between the shocks is greater than ΔX , f_X is approximated by

$$\tilde{f}_{X} = (\tilde{f}_{52,K+1} - \tilde{f}_{51,K+1}) \\ / (\tilde{X}_{52,K+1} - \tilde{X}_{51,K+1})$$

(12)

If, however, the distance between the shocks is smaller than ΔX , the values at (N, K+1) are interpolated from the shock values.

When a discontinuity crosses a mesh line, the computation of this mesh point must be modified. In Fig. 4, an up-running shock is shown crossing the Nth mesh line between steps K and K+I. The usual interior point computation would require taking the one-sided difference between points (N+K, K) and (N,K) for level one. This procedure is no longer proper, because of the presence of the shock. We, therefore, proceed by evaluating points A and B as follows

$$f_{A} = F_{N-1,K} + \delta(\tilde{f}_{N-1,K=1} - f_{N-1,K})$$

$$f_{B} = f_{s,K} + \delta(\tilde{f}_{s,K+1} - f_{s,K})$$

$$(13)$$

where

$$\delta = (X_N - X_{SK}) / (\tilde{X}_{SK+1} - X_{SK})$$
(14)

then, the value of f for level one at (N, K+I) is

$$\tilde{f}_{N,K+I} = f_B + \underline{A}_B \left(f_B - f_A \right) \left(I - \delta \right) \Delta Z / \Delta X \tag{15}$$

Care must be taken in evaluating δ when $dX_1/dZ \approx 0$., since Eq. (14) becomes indeterminate. We therefore require that,

$$\left| \begin{array}{c} \frac{\mathrm{d}X_{x}}{\mathrm{d}Z} \end{array} \right| \Delta Z > 0.2\Delta X$$

otherwise, δ is set to one to give the desired result

$$\tilde{f}_{N,K+1} = \tilde{f}_{s,K+1}$$

A similar computation is performed when a down-running discontinuity crosses a mesh line.

V. Wall Computation

The compatibility equation, Eq. (4), is applied at the wall to evaluate P_Z

$$P_{Z} = -\partial P_{X}^{\pm} \frac{\gamma w^{2}}{\left[\gamma T(\beta + w^{2}\tau^{2})\right]^{\frac{1}{2}}} (\tau_{Z} + \lambda \tau_{X})$$
(16)

where

$$\lambda = X_z + \lambda \pm X$$

Here, the X derivatives are evaluated as three-point one-sided differences and

$$b_{z} = \begin{cases} b_{zz} \text{ for } \lambda^{-} \\ c_{zz} \text{ for } \lambda^{+} \end{cases}$$

7

Then, following the MacCormack algorithm, the value of the pressure at the wall, P_w , is given by

$$\tilde{P}_{w,K+1} = P_{w,K} + P_Z \Delta Z$$
$$P_{w,K+1} = \frac{1}{2} \left(P_{w,K} + \tilde{P}_{w,K+1} + \tilde{P}_Z \Delta Z \right)$$

With the pressure and entropy known at the wall, the temperature is obtained from Eq. (2), and the magnitude of the velocity from

$$\frac{u^2 + w^2}{2} + \frac{\gamma}{\gamma - l} T = \text{constant}$$
(17)

The two-velocity component can then be determined from the known wall slope and the vanishing of the velocity component normal to the wall.

VI. Shock Detection

Perhaps the most distinguishing feature of the problem we are considering, and certainly its most troublesome one, is the possible formation of shock waves. In this section, we will consider the procedure required to detect an incipient shock. There are two important features for any such procedure. First, the procedure must be reliable. Incipient shocks must be located at the proper place, but most important, the procedure must be able to interpret the physics of the problem accurately, and should not be so sensitive as to create nonexistent shocks. Secondly, because the procedure must be executed at every step. it must be very fast and require a minimum amount of data to make a reliable decision.

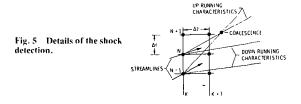
The procedure we have constructed, based on the concept of coalescing characteristics, requires only information on 2 adjacent mesh points, and we find it to be both reliable and fast. However, its empirical nature should be kept in mind. It is applied at the end of a computational step as follows:

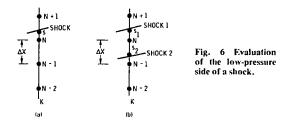
Assume that the flowfield is known at the Kth step (see Fig. 5). At each mesh point N and N-I, the up-running $(\lambda_{n}^{*}, \lambda_{i}^{*}, \nu_{-I})$ and downrunning $(\lambda_{n}^{*}, \lambda_{n-I})$ characteristics are evaluated using Eq. (5). If we now assume that the ΔZ intervals between steps (evaluated from the *CFL* condition)⁶ are essentially the same, then a quantity $\eta \pm$, which is the reciprocal of the number of steps needed for 2 characteristics to coalesce can be evaluated from

$$\eta \pm = (\lambda \hat{\kappa}_{-1} - \lambda \hat{\kappa}) \Delta z / \Delta r$$

A considerable number of numerical experiments indicate that a shock should be fitted into the calculation when η^* becomes greater than 0.22 (approximately 4 steps before the characteristics coalesce).

We have found that for a wide range of problems, if the detection of incipient shocks is delayed until $\eta^{\pm} < 0.22$), then the procedure becomes very sensitive and shocks are predicted unnecessarily.





The detected shock is assumed to have the average slope of the coalescing characteristics and to be located half way between r_N and r_{N-1} . With the initial shock location known, the initial flowfield variables on the low pressure side are linearly interpolated between the 2 mesh points bracketing the shock, and on the high pressure side they are linearly extrapolated using the 2 mesh points next to the shock on the high pressure side.

Shocks that are generated at the walls of the duct, because of slope discontinuities at the walls, are introduced from user supplied information on the wall geometries, and do not make use of the procedure described in this section.

VII. Shock Computation

The low-pressure side of a shock is computed using the same scheme described for the interior point computation, but using the X derivatives given in this section. In Fig. 6, assuming that mesh point N lies on the low-pressure side, the X derivatives at the shock for the predictor level are evaluated from

$$f_{X_{5}} = \left[\frac{2}{I+\epsilon} \left(f_{s,K} - f_{N-I,K} \right) + 0.5 f_{N-2,K} - \frac{(0.5+\epsilon)}{(I+2\epsilon)} f_{N,K} \right] / \Delta X$$
(18)

where

A

$$\epsilon = (X_{NK} - X_{NK}) / \Delta X \tag{19}$$

If only two mesh points are available, then the derivatives are evaluated from

$$f_{X_{s}} = \left[-\frac{2\epsilon - l}{l + \epsilon} f_{N-l,K} + \frac{3}{l + \epsilon} f_{s,K} - 2f_{N,K} \right] / \Delta X$$
(20)

with ϵ given by Eq. (19). The truncation error for Eq. (18) is of order ΔX^2 and for Eq. (20) it is

$$(\epsilon-I)\;\frac{\Delta X}{2}\,f_{XX_s}$$

It should be noted that Eqs. (18-20) are not the same formulas suggested by Moretti,^{2,4} particularly Eq. (18) has a lower truncation error than Moretti's counterpart.

If only 1 mesh point is available (Fig. 6b), then f_{X_s} is approximated by

$$f_{X_{5}} = (f_{5l,K} - f_{52,K}) / (X_{5l,K} - X_{52,K})$$
(21)

provided that $(X_{sl,K} - X_{s2,K})$ is greater than ΔX . If $(X_{sl,K} - X_{s2,K})$ is smaller than ΔX , then f_{X_s} is set to zero. For the corrector stage, the same derivatives are used but with f replaced by f.

If the low-pressure side lies on the other side of the shock, the derivatives that are used are the negatives of those given by Eq. (18) and Eq. (20), and ϵ is given by the negative of Eq. (19). The computation of the high-pressure side of the shock has been formulated using the compatibility equation, Eq. (4), on the characteristic reaching the shock on the high-pressure side along with Rankine-Hugoniot conditions. The details of this classical treatment of the shock can be found in Ref. 4.

The computation of the high-pressure side was the only computation requiring involved programing logic. The main difficulty is the determination of the origin of the characteristic reaching the shock in the immediate vicinity of other discontinuities. It should be, however, possible to avoid these difficulties by computing the high-pressure side with the method suggested by Kentzer.⁷

VIII. Contact Surface Computation[†]

To compute a contact surface, we exploit the fact that both pressure and streamline slope are continuous across a contact surface. Eqs. (3a, b), which involve derivatives of P and τ only, are thus recast in terms of the computational coordinates (X,Z), and are integrated using the MacCormack scheme where the X derivatives are evaluated using Eq. (20). In the predictor level, Eq. (20) is applied on the high Mach number side of the contact surface. In the corrector level, it is applied on the low Mach number side. This choice is arbitrary, and the same results are obtained if the order is reversed.

With the pressure and entropy known at the contact surface, the magnitude of the velocity follows from Eq. (17), and the 2 velocity components can be determined from the evaluated streamline slope.

IX. Interactions

The interactions of the discontinuities with the walls of the duct and with each other are computed by locally exact solutions. The details of these calculations can be found in most text books in fluid mechanics, for example, Ref. 8, and will not be discussed here.

X. Elimination of Weak Discontinuity

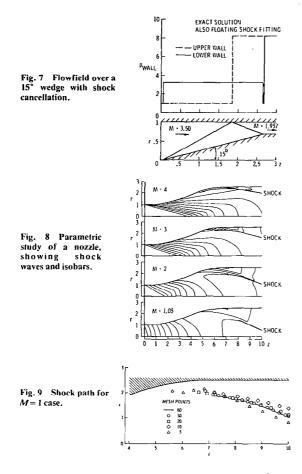
Discontinuities with a Mach number difference across the discontinity of less than 1% are automatically eliminated from the computation. This feature is not a requirement of the technique. It has, however, been incorporated, because for most practical applications, tracing these weak discontinuities is not necessary. It speeds up the computation, and it avoids computing degenerate discontinuities (i.e. the contact surface formed by the interaction of 2 shock waves of equal strength).

XI. Results

The results of several calculations performed with the floating shock fitting technique are presented to demonstrate the accuracy and versatility of the technique. All the computations were done on a CDC 6600 computer. The core required to run the code is under 55000₈ locations.

The flowfield produced by a uniform flow impinging on a wedge is often used to validate numerical calculations. The problem is a trivial one, consisting of uniform properties in front and behind the shock wave. The exact solution for a freestream Mach number of 3.50 which is initially compressed by a 15° wedge and then expanded, such as to exactly cancel the shock wave generated by the wedge, is shown in Fig. 7. In the figure, the lower wall pressure distribution shown corresponds to the case where the expansion occurs a very short distance before the shock reflects from the wall. The exact solution is obtained with the floating shock fitting

[†]We have recently experienced some difficulties with this procedure when the contact surface is next to a strong expansion fan. The problems have been overcome by reformulating the contact surface computation along the guide lines discussed in the shock computation.



technique using only 2 mesh points in less than 1 sec of computational time (the same results are obtained increasing the number of mesh points).

In Fig. 8, the results of a parametric study of the flowfield produced by a nozzle are presented. The initial Mach number was used as the parameter, allowing it to range from near 1 to 4. Both the isobars and shock-wave patterns are shown in the figure. For the Mach 4 case, most of the isobars downstream of the shock wave were not included in the figure to avoid overcrowding of lines. The total computational time for these 4 cases, using 30 mesh points, was approximately 25 sec. Figures 9 and 10 give some further details in the calculation of the Mach 2 case. In these 2 figures the results obtained with 5, 10, 20, 30, and 60 points are shown. Figure 9 shows the calculated shock-wave path, while Fig. 10 shows the upper wall pressure distribution. The results show that a good qualitative result for this more complicated problem can be obtained with as few as 5 mesh points, and that good quantitative results are obtained with only 10 mesh points.

It is very often argued that shock fitting is limited in its capability to handle flowfields with more than a single shock.^{9,11} In Refs. 3 and 12 this has been shown not to be the case for very complex three-dimensional steady, and onedimensional time-dependent flows. We hope to fill the existing gap for two-dimensional steady problems with Fig. 11 and also to demonstrate that floating shock fitting is a numerically stable technique even in multishocked flowfields. In Fig. 11, the geometry of NASA Langley's scramjet combustor model is simulated. The results shown are for a flow with an incoming Mach number equal to 3. inclined at 8° with the horizontal. The resulting flowfield con-

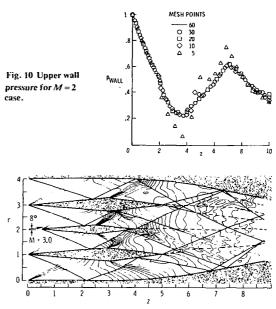


Fig. 11 Flowfield for a simulated scramjet, showing shock waves, vortex sheets and isobars.

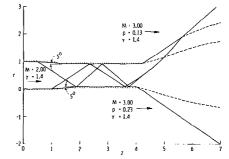


Fig. 12 Plume flowfield showing shock waves and vortex sheets.

sists of a complex pattern of interacting expansions and shock waves. In the figure, the shock-wave pattern is shown by the heavy lines. The contact surfaces are shown by the dashed lines, and isobars are shown by the light lines. Certain discontinuities are seen to terminate abruptly in the flowfield, while some shock interactions do not show the expected vortex sheet; this is a result of the automatic elimination of weak discontinuities previously discussed. The calculation was performed in 6 pieces, using a total of 120 mesh points and requiring approximately 2.2 min of computational time. The calculation shows approximately a dozen discontinuities, with some 40 interactions.

The technique has been extended to internal axisymmetric flows (with and without swirl), internal line source type flows, and to external two-dimensional flows. An example of the application to external flows is shown in Fig. 12, where a flow with an initial Mach number of 2.0 exhausts into a Mach 3.0 freestream. The lower wall ends at z=4.0, while the upper wall ends at z=4.2. The freestream conditions for the external flows are shown in the figure.

XII. Conclusions

Floating shock fitting for two-dimensional steady flows is an accurate and stable technique capable of solving very-complex problems with acceptable running times. A fully threedimensional code is planned for the near future.

References

重要力

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Appendix B INPUT GUIDE AND SAMPLE CASES

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Appendix B INPUT GUIDE AND SAMPLE CASES

DESCRIPTION OF INPUT DATA

ALL INPUT DATA IS READ FROM TAPE 5.

*THE FIRST INPUT CARD CONSISTS OF 11 INTEGERS.

THE FORMAT IS 1615

THESE INTEGERS ARE ID, KOUT, ICN, IBN, ISO, NA, ISY, JJ, ITY, ITP, IVI ITY IS USED TO INDICATE MERGING OF INTERNAL DUCTS, AND IS NOT RELATED TO JETS OR PLUMES.

- ID=0 INPUT CARD NO.3 IS NOT READ
- ID=1 INPUT CARD NO.3 IS READ
- KOUT = THE NO. OF STEPS TAKEN BEFORE A COMPLETE OUTPUT OF THE FLOW FIELD IS MADE
- ICN = THE TOTAL NO. OF SHARP CORNERS ON THE UPPER WALL ICN CANNOT EXCEED 10
- IBN = THE TOTAL NO. OF SHARP CORNERS ON THE LOWER WALL IBN CANNOT EXCEED 10
- ISO=0 NO ISOBAR OUTPUT WILL BE PROVIDED
- ISO=1 ISOBARS WILL BE COMPUTED AND PRINTED AT EVERY STEP
- NA ≈ INITIAL NUMBER OF MESH POINTS MINUS TWO NA CANNOT EXCEED 98 IN MOST CASES 20 POINTS ARE SUFFICIENT, AND ON THE AVERAGE THERE SHOULD BE ABOUT 5 POINTS BETWEEN DISCONTINUITIES.
- ISY=0 NO SYMMETRY IS ASSUMED IN THE COMPUTATION OF THE LOWER WALL
- ISY=1 SYMMETRY IS IMPOSED IN THE COMPUTATION OF THE LOWER WALL
- JJ=0 FOR TWO DIMENSIONAL FLOW
- JJ=1 FOR AXISYMMETRIC FLOW
- JJ=2 FOR LINE SOURCE FLOW
- ITY=0 A SINGLE DUCT CALCULATION
- ITY=1 SINGLE DUCT CALCULATION WITH DATA WRITTEN ON TAPE 7 FOR FUTURE MERGING RUN. CAN ALSO BE USED TO RESTART A CALCULATION
- ITY=2 DATA IS READ TO MERGE TWO OR MORE FLOWS, AND DATA IS WRITTEN ON TAPE 7 FOR FUTURE CONTINUATION
- ITY=3 DATA IS READ TO MERGE TWO OR MORE FLOWS
- ITP=0 NO CALCOMP PLOT IS GENERATED
- ITP=1 A CALCOMP PLOT IS GENERATED
- IVI=0 NO VISCOUS FORCES ARE COMPUTED
- IVI=1 SPALDING-CHI USED TO COMPUTE VISCOUS FORCES, THE REYNOLDS NO. WILL BE BASED ON THE WALL GEOMETRY LENGTH

- *INPUT CARD NO. 2 CONSISTS OF 5 INTEGERS THE FORMAT IS 1615
 NS = NO. OF CHEMICAL SPECIES
 NT = NO. OF POINTS IN THERMODYNAMIC TABLES
 NM = NO. OF CATALYTIC SPECIES
- NR = NO. OF REACTIONS IN MECHANISM
- IFROZE=0, FINITE RATE

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IFROZE=1, FROZEN FLOW

- *INPUT CARD NO. 3 IS READ ONLY IF ID=1 ITS PURPOSE IS SOLELY FOR IDENTIFICATION, A FREE FIELD FORMAT IS USED BETWEEN COLUMNS 6 THRU 55
- *INPUT CARD NO. 4 CONSISTS OF 5 FLOATING POINT NUMBERS THE FORMAT IS 8E10.4 THESE NUMBERS ARE ACH, GM, DIST, TOLER, SCALE
- ACH = INITIAL MACH NUMBER
- GM = RATIO OF SPECIFIC HEATS

DIST = DISTANCE TO THE END OF THE CALCULATION (CM)

- TOLER = A TOLERANCE PARAMETER TO INCREASE OR DECREASE THE NO. OF MESH POINTS. IT WORKS AS FOLLOWS: IF SET TO X THEN THE NO. OF MESH POINTS WILL BE INCREASED WHEN THE DISTANCE BETWEEN MESH POINTS IS X TIMES THE INITIAL DISTANCE BETWEEN MESH POINTS, AND MESH POINTS ARE DE-CREASED WHEN THE DISTANCE BETWEEN MESH POINTS IS 1./X TIMES THE INITIAL DISTANCE BETWEEN MESH POINTS. SETTING TOLER TO A LARGE NUMBER (SAY 1000.) HAS THE EFFECT OF NOT CHANGING THE NUMBER OF MESH POINTS.
- SCALE = IS A SCALE FACTOR FOR CALCOMP PLOTS NEEDED ONLY IF ITP=1 BOTH THE R AND Z COORDINATES ARE MULTIPLIED BY SCALE BEFORE THEY ARE PLOTTED.

*INPUT CARD NO. 5 CONSISTS OF 7 FLOATING POINT NUMBERS THE FORMAT IS 8E10.4

THE NUMBERS ARE ZEXIT, PINF, TINF, GMINF, ACHINF, ANGINF, RCINF THIS INFORMATION IS NEEDED IN CASE THE LOWER WALL OF THE DUCT OPENS TO THE FREE STREAM AT Z=ZEXIT. IF THIS IS NOT THE CASE THEN ZEXIT SHOULD BE SET TO A NUMBER GREATER THAN THE VALUE OF DIST, AND THE REMAINING SIX NUMBERS NEED NOT BE DEFINED.

ZEXIT = LOCATION OF EXIT PLANE (CM) PINF = FREE STREAM STATIC PRESSURE (ATM) TINF = FREE STREAM STATIC TEMPERATURE ([°]K) GMINF = NOT REQUIRED PRESENTLY ACHINF = FREE STREAM MACH NUMBER ANGINF = FREE STREAM INCLINATION IN DEGREES, MEASURED POSITIVE COUNTERCLOCKWISE FROM THE Z AXIS

RCINF = NOT REQUIRED PRESENTLY

INPUT CARD NO.6 (REQUIRED ONLY IF ZEXIT. LE. DIST) THE FORMAT IS 8E10.4 THIS CARD SUPPLIES THE SPECIES DISTRIBUTION IN THE EXTERNAL STREAM IN THE SAME ORDER AS IN THE MAIN STREAM^{}

ALFA(1, 1), FIRST SPECIES IN THERMO DATA TABLES ALFA(2, 1), SECOND SPECIES IN THERMO DATA TABLES ALFA(N, 1), N-TH SPECIES IN THERMO DATA TABLES

*****INPUT CARD NO. 7 IS THE SAME AS CARD NO. 5, BUT FOR THE CASE WHEN THE UPPER WALL OF THE DUCT OPENS TO FREE STREAM.

*INPUT CARD NO.8 IS THE SAME AS CARD NO.6 BUT FOR THE CASE WHEN THE UPPER WALL OF THE DUCT OPENS TO FREE STREAM.

*INPUT CARD NO. 9 CONSISTS OF ICN FLOATING POINT NUMBERS, IF ICN=0 THE INPUT CARD IS NOT NEEDED. THESE NUMBERS ARE STORED IN ARRAY ZIC(N). THE FORMAT IS 8E10.4

ZIC(N) = LOCATION OF THE NTH CORNER ON THE UPPER WALL (CM)

*INPUT CARD NO. 10 CONSISTS OF ICN FLOATING POINT NUMBERS, IF ICN=0 THE INPUT CARD IS NOT NEEDED' THESE NUMBERS ARE STORED IN ARRAY AIC(N). THE FORMAT IS 8E10.4

AIC(N) = THE CHANGE IN ANGLE IN DEGREES AT THE NTH UPPER WALL CORNER. AIC IS DEFINED POSITIVE IF THE CORNER IS A COM-PRESSION AND NEGATIVE IF AN EXPANSION. NOTE THAT IT IS A CHANGE IN ANGLE AND NOT THE NEW WALL ANGLE.

*INPUT CARD NO.11 IS THE SAME AS CARD NO.6, BUT FOR THE LOWER WALL. THE DATA IS STORED IN ARRAY ZIB(N), THE INPUT CARD IS NOT NEEDED IF IBN=0.

*INPUT CARD NO. 12 IS THE SAME AS CARD NO. 7, BUT FOR THE LOWER WALL. THE DATA IS STORED IN ARRAY AIB(N), THE INPUT CARD IS NOT NEEDED IF IBN=0. AIB IS DEFINED IN THE SAME WAY AS AIC.

*INPUT CARD NO. 13 CONSISTS OF 6 FLOATING POINT NUMBERS THIS CARD IS NOT NEEDED IF IVI=0 THE FORMAT IS 8E10.4

ATWTI(1) = NONDIMENSIONAL LOWER WALL TEMPERATURE ATWTI(2) = NONDIMENSIONAL UPPER WALL TEMPERATURE XL(1) = LOWER WALL VIRTUAL ORIGIN FOR SPALDING-CHI (GREATER THAN 0.) XL(2) = UPPER WALL VIRTUAL ORIGIN FOR SPALDING-CHI (GREATER THAN 0.) REI = REYNOLDS NO. PER UNIT LENGTH PRAND = PRANDTL NO.

^{*}IF PARTICULAR SPECIES DOES NOT EXIST ENTER ZERO IN THAT LOCATION.

*INPUT CARD NO. 14 INITIAL PRESSURE AND TEMPERATURE IN INTERNAL DUCT. THE FORMAT IS 8E10.4

PIN = INITIAL PRESSURE (ATM) TIN = INITIAL TEMPERATURE (^OK)

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***INPUT CARD(S) NO. 15 SPECIES DISTRIBUTION MOLE FRACTION IN INTERNAL DUCT.**

ALFA(1, 1), FIRST SPECIES IN THERMO DATA TABLES ALFA(2, 1), SECOND SPECIES IN THERMO DATA TABLES ALFA(N, 1), NTH SPECIES IN THERMO DATA TABLES

*INPUT CARD(S) NO. 16 SPECIES ID AND THERMODYNAMIC DATA THE FORMAT IS A6, 2E10.3

AID(I). ITH SPECIES IDENTIFICATION WTM(I). ITH SPECIES MOLECULAR WEIGHT (g/g-mole) HF(I), ITH SPECIES HEAT OF FORMATION (K-cal/mole) TTB(J), JTH TEMPERATURE POINT IN TABLES (K) CPTB(1, J) SPECIFIC HEAT FOR ITH SPECIES AT JTH TEMPERATURE POINT (cal/mole-K) STB(K, J) ENTROPY FOR ITH SPECIES AT JTH TEMPERATURE POINT (cal/mole-K) HTB(I, J) ENTHALPY FOR ITH SPECIES AT JTH TEMPERATURE POINT (K-cal/mole) *INPUT CARD(S) NO. 17 CATALYTIC SPECIES DATA (OPTIONAL) INPUT ONLY IF NM=0 THE FORMAT IS A6/(16E5.2)IF NM $\neq 0$, INPUT THE FOLLOWING AID(J), NAME OF JTH CATALYTIC SPECIES WF(J,K), WEIGHTING FACTOR OF ITH SPECIES FOR JTH CATALYTIC SPECIES

AID(J), NAME OF JTH CATALYTIC SPECIES WF(J, I), WEIGHTING FACTOR OF ITH SPECIES FOR JTH CATALYTIC SPECIES

*INPUT CARD NO. 18 NAME OF REACTION SET THE FORMAT IS 13A6, A2

*INPUT CARD NO. 19 REACTION RATE DATA (OPTIONAL) INPUT ONLY IF IFROZE IS ZERO THE FORMAT IS A6, 1X, A6, 1X, A6, 1X, A6, 1X, A6, 1X, A6, 7X, 12, 11, E8.2, F5.2, F10.1, 1F6.2 AID(J), SPECIES (NAME) APPEARING IN ITH REACTION IRR(I), REACTION TYPE IRT(I), RATE CONSTANT TYPE RCT(I, N), RATE CONSTANT COEFFICIENTS N = 1,4

FOR MOST RUNS, WITH THE EXCEPTION OF CASES WHERE ITY=2 OR ITY=3, THIS IS ALL THE INFORMATION REQUIRED AS INPUT DATA. CASES WHERE ITY=2 OR ITY=3 WILL BE DISCUSSED LATER. LET US CONSIDER NOW THE DEFINITION OF THE WALL GEOMETRIES.

THE LOWER WALL GEOMETRY SHOULD BE DEFINED IN SUBROUTINE GEOMB. THE USER DEFINES THE GEOMETRY BY GIVING R OF THE WALL AS A FUNCTION OF THE AXIAL COORDINATE Z. THE CODE ASSUMES THAT THE INITIAL STATION IS AT Z=0. IF IT IS NECESSARY TO START THE CALCULATION AT SOME OTHER VALUE OF Z, THIS CAN BE DONE BY REDEFINING Z IN THE MAIN PROGRAM. IN ADDITION THE FIRST DERIVATIVE, RZ, AND THE SECOND DERIVATIVE, RZZ, ARE ALSO REQUIRED.

THE UPPER WALL GEOMETRY SHOULD BE DEFINED IN SUBROUTINE GEOMC. THE GEOMETRY IS DEFINED IN THE SAME MANNER AS DE-SCRIBED ABOVE. A SAMPLE UPPER GEOMETRY FOR A DUCT WITH AN INITIAL HEIGHT OF ONE UNIT, A 10. DEGREE COMPRESSION AT Z=.5 AND A 6 DEGREE EXPANSION AT Z=1. IS:

R=1. \$ RZ=0. \$ RZZ=0. \$ IF (Z.LT.0.5) RETURN R=1. -.17633*(Z-.5) \$ RZ=-.17633 \$ RZZ=0. IF (Z.LT.1.) RETURN R=.911835+.1051*(Z-1.) \$ RZ=.1051 \$ RZZ=0. \$ RETURN

CARE SHOULD BE EXERCISED WHEN DEFINING WALL GEOMETRIES, AS THE MOST FREQUENT CAUSE OF PROBLEMS THAT USERS HAVE EX-PERIENCED WITH THIS PROGRAM IS USUALLY RELATED TO IMPROPER GEOMETRY DEFINITION.

The pre-exponential factor, N = 1, has units of (cm-particle-sec). The activation energy, N = 3, has the units (cal/mol).

SAMPLE CASE FOR FINITE RATE CHEMISTRY

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000017		700.		31-632	1.995	800.		32.296	2:493
000018	000	900.	4.968	32.881	2.990	1000.	4.968	33.404	3.487
000019	000	1200.	4.968	34.310	4.481	1460.	4.968	35.075	5.474
000020	000	1600.	4.968	35.739	6.468	1800.	4 • 96 8	36.325	7.461
000021	000	2000.	4.968	36.848	8.455	2200.	4.968	37.322	9.449
000022	000	2400.	4.968	37.754	10.442	2600.	4.968	38.152	11.436
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000026	000	200.	6.894	31.251	.013	400.	6.975	33.247	.707
000027	000	500.	6.993	34.806	1.406	600.	7.009	36.082	2.106
000028	000	709.	7.036	37.165	2.808	800.	7.087	38.107	3.514
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000032	000	2000.	8.195	45.004	12.651	2200.	8.358	45.793	14.307
000033	000	2400.	8.506	46.527	15.993	2600.	8.639	47.213	17.708
000034	000	2800.	8.757	47.857	19.448	3000.	8.859	48.465	21.210
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000038	000	500.	8.415	49.334	1.654	600.	8.676	50.891	2.509
000039	000		8.954	52.249	3.390	800.	9.246	53.464	4.300
000040	000	900.	9.547	54.570	5.240	1000.	9.851	55.592	6.209
-000041			10:444	57-441				··59:092-	
000042	000	1600.	11.462	60.591	12.630	1800.	11.869	61.965	14.964
000043	···· 000 ···	2000.	12.214	63.234	17.373	2200.	12.505	64.412	19.486
000044	000	2400.	12.753	65.511	22.372	2600.	12,965	66.541	24.945
-000045		2800.	13-146	67.508	27.556	3000.	13.304	68.421	30.201
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000050	000	700.	5.029	42.831	2.048	800.	5.015	43.501	2.550
000051	000		5.006		3.052	1000.	4.999	44.819	
000052	000	1200.	4.990	45.529	4.551	1400.	4.984	46.298	5.548
000053			4.981	46.963	6.544	1800 ·		47.550	7.540
000054	000	2000.	4.978	48.074	8.536	2200.	4.979	48.549	9.532
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000059 000060	000 000	200. 500.	7.05			600.	7.057	48.837	2.137
	000	700.	7.05			800.	7.150	50.877	3.556
000061							7.332	52.491	5.003
000062	000	900.	7.23			1000.			
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000064	000	1600.	7.96			1800.			
000065	800	2000.	8.280			2200.	8.415	58.686	13.681
600066	000	2400.	8.520			2600.	8.622	60.110	17.929
000067	000	2800.	8.700			3000.	8.780	61.355	21.411
000068	000	3200.	8.840		4 23+174	3400.	8.905	62.462	24.949
000069	000			0.0					
000070	600	200.	7.02			460.	7.196	51.091	0.724
000071	000	500.	7.43			600.	7.670	54.098	2.210
000972	000	700.	7.88			800.	8.063	56.361	3.786
660073	600	900.	8.21			1000.	8.336	58.192	5.427
000074	000	1200.	8.52			1400.	8.674	61.055	8.835
000075	000	1600.	8 • 8 0 1			1800.	R.916	63.265	12.354
000076	000	2000.	9.02			2200.	9.139	65.075	15.966
770066	000	2400.	9.248			2600.	9.354	66.620	19.664
000078	000	2800.	9.45	67.31	21.545	3000.	9.551	67.973	23.446
000079	aco	3200.	9.64	68.592	25.365	3400.	9.723	69.179	27.302
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000061	000	200.	8.34	7 54.434	0.015	400.	8.907	56.910	0.877
000082	000	500.	9.470	58.960	1.797	600.	9.980	68.734	2.771
000083	000	700.	10.40	62.305	5 3.791	800.	19.769	63.719	4.850
000064	000	900.	11.08	65,006	5.943	1900.	11.365	66.189	7.066
000085	000	1200.	11.83	68.304	9.387	1400.	17.197	70.156	11.791
000066	000	1600.	12.485	5 71.804	14.761	1800.	12+714	73.288	16.782
000067	000	2000.	12.89	5 74.638	3 19.343	2260.	13.041	75.874	21.937
000068	000	2400.	13.160	77.014	4 24.558	2600.	13.256	78.371	27.200
000089	000	2800.	13.330		29.859	3000.	17.403	79.979	32.534
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000100	000	н +0		=H20 +M1			2 6.10-26 2.0	0.0	
000101	000	н +0		=H02 +M3			3 5.5 -33	1000	
000102	000	0 +0	+M1	=02 +M1			4 3.80-30 1.0	-340.0	
000103	000	0H +H		=02 +0			4 1.40-14-1.0	-7000.0	
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000166	000						-		
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000108	000	H02 +H	-	=0H +0H			3 4.2 -10	- 1900	
000109	000	H02 +H	2	=H20 +0H			3 1.2 -12	- 18700	
000110	000	H02 +0		=0H +02			3 8.0 -11	- 1000	
000111	000	H02 +01	4	=H20 +02	5	1	3 8.3 -11	- 1000	• •

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END ELT.

aADD,P DATA

axot seagull

PROGRAM SEAGULL SUPERSONIC INVISCID FLOW ANALYSIS NASA LANGLEY RESEARCH CENTER RELEASED 02/01/76 LAST MODIFICATION 02/25/77

TWO DIMENSIONAL FLOW TWO DIMENSIONAL TEST CASE WITH SHOCK + INTERACTI OUTPUT AT EVERY 1 STEPS GAMMA= .1230+01 DISTANCE= .9000+01 TOLERANCE= .1000+02 SCALE ICN=1 IBN=1 ISO=G ISY=0 ITY=0 ITP=0 IVI=0 .2000+02 DEGREES BREAK IN THE UPPER WALL AT Z= .1500+01 .1000+02 DEGREES PREAK IN THE LOWER WALL AT Z= .1000+01

INITIAL CONDITIONS

TEMPERATURE (DEG. VELOCITY (M/S) PRESSURE (ATM) DENSITY (G/CC) MOLECULAR WT. GAMMA	KELVIN)	1.0000000+03 5.3167825+03 6.8050000-01 1.1281265-04 1.3602172+01 1.2846758+00
TOTAL ENTHALPY		5.7722905+11 2.1301588+09
MOLE FRACTION	н	3.4969999-02
MOLE FRACTION	HZ	2.3927000-01
MOLE FRACTION	H20	6.9387999-01
MOLE FRACTION	0	2.4700000-03
MOLE FRACTION	0 H	2.6460000-02
MOLE FRACTION	02	2.9400000-03
MOLE FRACTION	H02	0.0000000

FINITE RATE CHEMISTRY CALCULATIONS

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H2 - 02 C OMBUST ION KI NETICS

REACTIONS BEING CONSIDERED

KF=A+EXP(B/RT++H)/T++N

							A	N	B	M
1	н	+ н	+ M2	= H2	+ M2	22	1.016+18	1.00	.0	.00
	н	+ 0	+ M1	= OH	+ H1	21	3+628+15	.00	• 0	.00
-3	н	+ OH	+ M1	≈ H2O	+ M1	22	2 • 21 3+22	2.00	.0	.00
	H .	+ 02	+ M3	= H02	+ M3	23	1.995+15	.00	1000.0	.00
5	0	+ 0	+ M1	= 02	+ M1	24	1.379+18	1.00	-340.0	•00
	OH	+ н		= H2	+ 0	1 4	8.432+09	-1.00	-7000.0	•00
7	он	+ 0		= н	+ 02	1 1	2.409+13	.00	.0	.00
	OH	+ H2		= H2O	+ H	14	6.023+06	-2.00	-2900.0	.00
9	он	+ OH		= H20	+ 0	1 3	6.023+12	• 00	-1100.0	.00
10	H02	+ H		= H2	+ 02	1 3	2.530+13	.00	-700.0	.00
11	H02	+ н		= 0H	+ OH	1 3	2.530+14	• 00	-1900.0	.00
12	H02	+ H2		= H2O	+ OH	1 3	7.228+11	.00	-18700.0	.00
13	H02	+ 0		= OH	+ 02	1 3	4.819+13	.00	-1000.0	.00
· · · · · · · · · · · · · · · · · · ·	H02	+ 0H		= H2O	+ 02	1 3	4.999+13	.00	-1000.0	•00

CATALYTIC SPECIES BEING CONSIDERED

···· · · · ·			, 1.00 H2	, 6.00 H20	, 1.00 0	• 1.00 OH	, 1.90 02	¥ 1.00 H02	,
	MZ	=20.00 H	, 2.50 H2	,10.00 H20	, 1.00 0	, 1.00 OH	, 1.00 02	# 1.00 HO2	1
···· · ·· -··	M3	= 1.00 H	, 2.50 H2	,16.00 H20	, 1.00 0	, 1.00 OH	, 1.00 02	, 1.00 H02	•

K= 1 Z= .7000+00 DZ= .0000

UPPER SUM DI FORCE	FORCES	FR= .0000 .0000 IONAL PLANE=	FZ=		.0000 .0000 .0000 .0000 0000 IS= .0000	VFR= VFR=	.0000 VFZ= .0000 VFZ= .0000	•0000 •0000 •0000	VM= .000 VM= .000 .000	Ō
N 0 21	• 1000+01	• 1000+01	ידי 2000	₩ •6801+01	T • 1000+81	S •0000	M •6030+01	THE -	RH0 • 1000+01	HTOT •1538+01
C 20 0 19	•9474+00 •8947+00	+1000+01	.0000	+6801+01 +6801+01	•1000+01 •1000+01	.0000	+6030+01 +6030+01	.0000	•1000+01 •1000+01	<pre>.1538+01 .1538+01</pre>
Ο <u>18</u> ί 17	•8421+00 •7895+00	+1000+01 +1006+01	.0000	•6801+01 •6801+01	•1000+01 •1000+01	•0000 •0000	•6000+01 •6000+01	.0000	•1000+01 •1000+01	•1538*01 •1538+01
0 16 0 15	•7368+00 •6842+00	+1000+01 +1000+01	.0000 .0000	.6801+01 .6801+01	•1090+01 •1000+01	.0000 .0000	•5000+01 •6000+01	•0000 •0000	•1000+01 •1000+01	•1538+01 •1538+01
	•6316+00 •5789+00		0000.	.6801+01 .6801+01	•1000+01 •1000+01	.0000	.6000+01 .6000+01	.0000	•1000+01 •1000+01	<pre>.1538+01 .1538+01</pre>
C 12 C 11	• 4737+00 • 4737+00	•1000+01 •1000+01 •1000+01	0000 .0000 .0000	•6801+01 •6801+01 •6801+01	•1000+01 •1000+01 •1000+01	•0000 •0000 •0000	•6030+01 •6030+01 •6030+01	.0000 .0000 .0000	•1000+01 •1000+01 •1000+01	+1538+01 +1538+01 +1538+01
0 9	•3684+00	+1000+01	.0000	.6801+01	•1000+01	•0000	.6000+01	.0000	• 1000+01	•1538+01

0.	18	.3158+00	.1000+01	.0000	•68C1+O1	.1000+01	.0000	+6030+01	.0000	+1000+01	1538+01
۵	7	.2632+00	1000+01	.0000	.6801+01	.1000+01	•0000	+6000+01	•0000	1000+01	1538+01
0	6	+2105+0C	.1000+01	.0000	+6801+01	1000+01	•0000	•6000+C1	.0000	1000+01	1538+01
G	5	1579+00	.1000+01	.9000	.68 01+01	1000+01	.0000	.6030+G1	.0300	.1000+01	+1538+01
۵	4	+1053+00	.1000+01	.0000	.6801+01	1000+01	.0000	+6030+C1	•0000	1000+01	+1538+91
0	3	•5263-01	1000+01	.0000	.6801+01	.1000+01	.0000	.6000+01	.0000	• 1000+01	+1538+01
-0-	Z	.0000	" . 1000+01	.0000	.6801+01	.1000+01	.0000	.6000+01	.0000	.1000+01	1538+01

MOLE FRACTIONS

PT	R (N)	н	H2	H20	0	0H	02	HOZ	MOL WT	GANNA
21	1.00000	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
20	.94737	3.49704-02	2.39272-01	6.93987-01	2.47002-03	2.64603-02	2.94603-03	0.00000	9.99990-01	1.28467+00
19	.89474	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
- 18-	.84211	3.49704-02	2.37272-01	6.93887-01	2.47002-03	2.64603-02	2,94003-03	0.00000	9.99998-01	1,28467+00
17	.78947	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
16-	.73689	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.0000	9.99990-01	1.28467+00
15	•68421	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64607-02	2.94003-03	0.00000	9.99990-01	1.28467+00
- 14	.63158	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.0000	9 . 99 990-01	1.28467+00
13	.57895	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
- 12	.52632	3.49704-02	2.39272-01	6.93887-01	2 47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
11	• 47368	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
10	+42105	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467#00
9	.36842	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
8 .	-31579	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
7	.26316	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
	21053	3.49704-02	2.39272-01	6,93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
5	.15789	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
4	.10526	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.80000	9.99990-01	1.28467+00
3	.05263	3.49704-02	2.39272-01	6.93887-01	2.47002-03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28467+00
2	.00000	3.49704-02	2.39272-01	6.93887-01	2.47002~03	2.64603-02	2.94003-03	0.00000	9.99990-01	1.28*67.+00

SAMPLE CASE FOR VIBRATIONAL NON-EQUILIBRIUM

000001	000	01-19:12:01 axot seagu							
000002	000	1 100		1 19	•				
000003	000	12 24		• ••••♥ ··•••• 5 0					
000004	000			-	***				
000005	000	4.0	1.4		10.0	ن • ن			
000006	000	434.							
000007	000	434.							
000008	000	1.0							
000009	000	-30.0							
000010	000	C+998							
000011	000	-30.							
000012	000	2.0	2000.	4340.					
000013	000	0.00322	0.00162	0.00876	0.01083	0.01018	0.00873	0.00393	
000014	000	0.37433	0.07024	0.05909	0.02363	0.42544			
000015	000	C02100 44		0.081	0102000	51.25.1			
000016	000	0.	6.95		-2.074	50.	6.95	5 37.932	-1.725
000017	000	100.	6.95				6.95		-1.030
000018	000	200.	6.95				6.95		-0.334
000019	000	300.	6.95				6.95		0.709
000020	000	500.	6.95				6.95		2.100
000021	000	700.	6.95				6.95		3.491
000022	000	900.	6.95				6.95		4.882
000023	000	1200.	6.95				6.95		7.664
000024	000	1600.	6.95				6.95		10.446
000025	000	2000	6.95				6.95		13.229
000026	000	2400.	6.95				6.95		16.011
000027	000	2800.	6.95				6.95		18.793
000028	000			7.331			••••		
000029	000	0.	6.95		2 -2.074	4 50.	6.95	5 37.932	-1.725
000030	000	100.	6.95				6.95	-	-1.030
000031	000	200.	6.95				6.95		-0.334
000032	000	300.	6.95				6.95		0.709
000033	000	500.	6.95				6.95		2.100
000034	000	700.	6.95				6.95		3.491
000035	000	900.	6.95				6.95		4.882
000036	000	1200.	6.95				6.95		7.664
000037	000	1600.	6.95				6.95		10.446
000038	000	2000.	6.95				6.95		13.229
000039	000	2400.	6.95				6.95		16.011
000040	000	2800.	6.95				6.95		18.793
000041	000			4.051					

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000042	ÜÖ B	0.	6.955	37.932	-2.074	50.	6.955	37.932	-1.725
000043	000	103.	6.955	42.753	-1.378	150.	6.955	45.574	-1.030
000044	000	200.	6.955	47.575	-0.682	250.	6.955	49.127	-0.334
000045	000	300.	6.955	50.395	0.013	400.	6.955	52.396	0.709
000046	000	500.	6.955	53.948	1.404	600.	6.955	55.216	2.100
000047	000	700.	6.955	56.286	2.796	800.	6.955	57.217	3.491
000048	000	900.	6.955	58.036	4.187	1000.	6.955	58.769	4.882
UDC049	600	1200.	6.955	60.037	6.273	1400.	6.955	61.109	7.664
000050	000	1600.	6.955	62.038	9.055	1800.	6.955	62.857	10.446
000051	0 00	2000.	6.955	63.590	11.838	2200.	6.955	64.253	13.229
000052	000	2400.	6.955	64.855	14.620	2600.	6.955	65.415	16.011
000053	000	2800.	6.955	65.930	17.402	3000.	6.955	66.410	18.793
000054	000	CO2110 44.01	-88.1					•	
000055	000	D •	6.955	39.31	-2.074	50.	6.955	39.31	-1.725
000056	000	100.	6.955	44.131	-1.378	150.	6.955	46.951	-1.030
000057	000	200.	6.955	48.952	-0.682	250.	6.955	50.504	-0.334
000058	000	300.	6.955	51.772	0.013	400.	6.955	53.773	0.709
000059	000	500.	6.955	55.325	1.404	600.	6.955	56.593	2.100
000060	0 0 0	7úg.	6.955	57.665	2.796	800.	-6.955	58.594	3.491
000061	000	960.	6.955	59.413	4.187	1000.	6.955	60.146	4.882
000062	000	1200.	6.955	61.414	6.273	1400.	6.955	62.487	7.664
0000ė3	000	1600.	6.955	63.415	9.055	1800.	6.955	64.235	10.446
000054	000	2000.	6.955	64.967	11.838	2200.	6.955	65.63	13.229
000065	000	2400.	6.955	66.235	14.620	2600.	6.955	66.792	16.011
000066	000	2800.	6.955	67.308	17.402	3000.	6.955	67.788	18.793
000067	000	C02010 44.01	-92.1						
000068	UD 0	Ο.	6.955	39.31	-2.074	50.	6.955	39.31	-1.725
000069	000	100.	6.955	44.131	-1.378	150.	6.955	46.951	-1.030
000076	000	200.	6.955	48.952	-0.682	250.	6.955	50.504	-0.334
000071	000	300.	6.955	51.772	0.013	400.	6.955	53.773	0.709
000072	000	500.	6.955	55.325	1.404	600.	6.955	56.593	2.100
000073	000	700.	6.955	57.665	2.796	800.	6.955	58.594	3.491
000074	000	900.	6.955	59.413	4.187	1600.	6.955	60.146	4.882
000075	000	1200.	6.955	61.414	6.273	1400.	6.955	62.487	7.664
000076	ບບົບ	1600.	6,955	63.415	9.055	1800.	6.955	64.235	10.446
000077	000	2000.	6.955	64.967	11.838	2200.	6.955	65.63	13.229
000078	000	2400.	6.955	66.235	14.620	2600.	6.955	66.792	16.011
000079	000	2800.0	6.955	67.308	17.402	3000.0	6.955	67.788	18.793
000080	000	C02020 44.01	-90.2		1	50.	6.955	40.115	-1.725
000081	600	0.	6.955	40.115	-2.074		6.955	47.757	-1.030
000082	000	100.	6.955	44.936	-1.378	150.		51.310	-0.334
000083	000	200.	6.955	49.758	-0.682	250.	6.955 6.955	54.579	0.709
000034	000 000	300.	6.955	52.578	0.013 1.404	400. 600.	6.955	57.399	2.100
000085	600	500. 700	6.955 6.055	56.131	2.796	800.	6.955	59.400	3.491
000066		7UD.	6.955	58.471 60.219	4.187	1000.	6.955	60.952	4.882
0000x7	000	900.	6.955						7.664
880000	000	1200.	6.955	62.220	6.273	1400.	6.955	63.292 64.040	10.446
000089	000 000	1600.	6.955	64.221 65.773	9.055 11.838	1860. 2200.	6.955 6.955	66.436	13.229
00000	000	2000.	6.955			2600.	6.955	67.598	16.011
000091	000	2400.	6.955	66.041	14.620	3000.	6.955	68.593	18.793
000092	000	2800. 002030 00 01	6.955	68.113	17.402	2000.	0.733	000073	100173
000093	000	CO2O3O 44.01	-88.5	121					

000094	600	0.	6.955	40.687	-2.074	50.	6.955	40.687	-1.725	
000095	000	100.	6.955	45.508	-1.378	150.	6.955	48.329	-1.030	
000096	6 0 0	200.	6.955	50.329	-0.632	250.	6.955	51.882	-0.334	
000097	000	300.	6.955	53.150	0.013	400.	6.955	55.151	0.709	
000098	200	500.	6.955	56.703	1.404	600.	6.955	57.971	2.100	
000099	000	700.	6.955	59.043	2.796	800.	6.955	59.972	3.491	
000100	000	900.	6.955	60.791	4.187	1000.	6.955	61.524	4.882	
000101	000	1200.	6.955	62.792	6.273	1400.	6.955	63.664	7.664	
	000	1600.	6.955	64.793	9.055	1800.	6.955	65.612	10.446	
000102		2000.	6.955	66.345	11.838	2200.	6.955	67.008	13.229	
000103	000			67.613	14.620	2600.	6.955	68.170	16.011	
000104	000	2400.	6.955						18.793	
000105	000	2800.	6.955	68.685	17.402	3000.	6.955	69.165	10.175	
000106	000	N2(D) 28.		3.798	71.4			77 770		
000107	000	Ũ.	6.955	33.339	-2.075	50.	6.955	33.339	-1.727	
000108	000	100.	6.956	38.160	-1.379	150.	6.956	40.981	-1.031	
000109	000	200.	6.957	42.982	-0.683	250.	6.957	44.535	-0.335	
300110	000	300.	6.958	45.803	0.013	400.	6.959	47.805	0.709	
000111	000	5ິນວ∙	6.960	49.357	1.405	600.	6.962	50.6Z7	2.101	
000112	000	700.	6.965	51.700	2.797	800.	6.968	52.631	3.494	
000113	000	900.	6.972	53.451	4.190	1000.	6.975	54.186	4.888	
000114	000	1200.	6.983	55.458	6.284	1400.	6.991	56.535	7.681	
000115	0 0 6	1600.	6.999	57.470	8.963	1800.	7.008	58.294	10.481	
000116	000	2000.	7.015	59.033	11.883	2200.	7.022	59.702	13.287	
000117	000	2400.	7.031	60.313	14.692	2600.	7.038	60.876	16.099	
000118	000	2800.	7.045	61.399	17.507	3000.	7.052	61.885	18.917	
000119	000	N2(1) 28.	016 6.65	55 3.798	71.4					
000120	000	0.	6.955	33.339	-2.075	50.	6.955	33.339	-1.727	
000121	000	100.	6.956	38.160	-1.379	150.	6.956	40.981	-1.031	
000122	600	200.	6.957	42.982	-0.683	250.	6.957	44.535	-0.335	
000123	000	300.	6.958	45.803	0.013	400.	6.959	47.805	0.709	
000124	000	500.	6.960	49.357	1.405	600.	6.962	50.627	2.101	
000125	000	700.	6.965	51.700	2.797	800.	6.968	52.631	3.494	
000126	000	900.	6.972	53.451	4.190	1000.	6.975	54.186	4.888	
000127	000	1200.	6.983	55.458	6.284	1400.	6.991	56.535	7.681	
000128	000	1600.	6.999	57.470	8.963					
000129	000	2000.				1800.	7.008	58.294	10.481	
000130	000	2400.	7.015	59.033	11.883	2200.	7.022	59.702	13.287	
			7.031	60.313	14.692	2600.	7.038	60.876	16.099	
000131	000	2800.	7.045	61.399	17.507	3000.	7.052	61.885	18.917	
000132	000		9988 0.0							
000133	000	0.	6.958	36.572	-2.075	50.	6.958	36.572	-1.729	
000134	000	100.	6.958	41.395	-1.381	150.	6.960	44.216	-1.033	
000135	000	200.	6.961	46.218	-0.685	250.	6.992	47.769	-0.337	
000136	000	300.	7.023	49.047	0.013	400.	7.196	51.091	0.724	
000137	000	500.	7.431	52.722	1.455	600.	7.670	54,098	2.210	
000138	000	700.	7.883	55.297	2.988	.006	8.063	56.361	3.786	
000139	000	900.	8.212	57.320	4.600	1000.	8.336	58.192	5.427	
000140	000	1200.	8.527	59.729	7.114	1400.	9.674	61.055	8.835	
000141	000	1600.	8.900	62.222	10.583	1800.	8.916	63.265	12.354	
000142	000	2000.	9.629	64.210	14.149	2263.	9.139	65.076	15.966	
000143	000	2400.	9.248	65.876	17.804	2600.	9.354	66.620	19.664	
000143		3860	9.455	67.317	21.545	3000.	9.551	67.973	23.446	
000144 000145	000	2800.	6 -57,79	01011	C10342					

,

000146	000	0.0	ü•0	0.0	-2.	367	50.	3.9805	18.	198	-1.974
000147	000	100.	7.961	36.396		581	150.	7.965			-1.18
000148	000	200	7.969	41.916		784	250.	7.998		-	384
000149	600	300.	8.027	45.155		015	400.	8.186		483	.825
000150	000	500.	8.415	49.334		654	600.	8.676		891	2.509
000151	000	700.	8.954	52.249		390	800.	9.246		464	4.300
000152	000	900.0	9.547	54.570		240	1000.0	9.851		5.591	6.209
000152	000	1200.0	10.444	57.440		240	1400.0	10.987			10.384
000153	000	1603.0	11.462	60.591		630	1800.0	11.869	-		14.964
			12.214	63.234		373	2200.0	12.505			19.846
000155	000	2000.0					2600.0	12.965			24.945
000156	000	2400.0	12.753	65.511		372					
000157	000	2800.0	13.146	67.508	27.	556	3000.0	13.304	6	8.420	30.201
000158	600	HE	4.003	0.							
000159	000	Ο.			255	-1.481			968	21.255	
000160	000	103.			698	-0.984			968	26.128	
000161	000	200.			142	-0.488			968	29.048	
000162	000	300.			156	0.009			968	31.586	
000163	000	500.			694	1.003			968	33.600	
000164	000	700.	4.	968 34.	366	1.996	800.	4.	968	35.029	
000165	000	900.	4.	968 35.	614	2.990	1000.	4.	968	36.138	3.487
000166	000	1200.	4.	968 37.	044	4.481	1400.,	4.	968	37.809	5.47#
000167	000	1600.	ц	968 38,	473	6.468	1800.	4 -	968	39.058	7.461
000168	000	2000.	-		581	8.455			968	40.055	
						10.442			968	40.885	
000169	000	2400.			487						
000170	000	2800.	4.	968 41	253	12.430	3000.	4.	968	41.596	13.423
000171	000	MI		- · · ·					- ·	•	
000172	000	1.0 1.0	1.0 1.	0 1.0 1.	U I•l	1 U.U	0.0 0.0	1.U O.	0 1.	U	
000173	000	M2		_						-	
000174	000		1.0 1.	n 1.0 1.	0 1.0	1.0	1.0 1.0	1.0 1.	5 1.	0	
000175	000	M 3									
000176	000	1.0 1.0	1.0 1.	0 1.0 1.	.0 1.0	0.5	0.5 0.6	1.0 0.	0 1.	0	
000177	UD0	M4									
000176	000	0.0 0.0	0.0 0.	0 0.0 0.	0 0.0	2.9	2.5 2.4	0.0 1.	0 0.	0	
000179	000	CC	D2 GDL RE	ACTION MOL	EL (A)	R-BREAT	HING) 19	0CT 197	6 26	REACTION	IS
000180	000	C02110+C0		=C02100				12 1.25-			
000181	000	C02D30+C0		=C02100				12 1.8 -	15-0.	5	
000182	000	C02030+C0		= C 0 2 0 2 0				12 3.1 -			
000183	000	C02100+C		=C02010				11 4.0 -		-	
800184	000	C02020+C0		=C02010				12 1.4 -		5	
000185	000	C02001+M		=C0211i				64 1.1 -			
000186	000	C02001+M4		=C0211				64 1.9 -			
000187	000	C02001+M		=02030				64 8.1 -			
								64 1.4 -			
000138	000	C02001+M4		=02030							
000189	000 0 0 0	C02110+M2		=0203				62 4.3 -			
000190	000	C02110+M2	2	=02026	3+82			64 4.5 -	21-4.	2 - 903	*

000191	000	C02110+M2	=C02020+M2	64 8.8 -20-2.5 - 4410.
000192	000	C02110+M2	=C02100+M2	64 8.6 -24-3.8 - 549.
000193	000	C02030+M2	=C02020+M2	64 9.3 -22-3.3 - 1230.
000194	000	CU2030+M2	=C02100+M2	64 1.1 -21-3.0 - 1060.
000195	000	C02100+M2	=C02020+M2	62 8.0 -18-1.5
000196	000	C02100+M3	=C02010+M3	64 5.6 -22-3.3 - 1480.
000197	000	C02100+HE	=C02010+HE	14 1.8 -21-3.0 843.
000198	000	C02020+M3	=C02010+M3	64 2+1 -21-3+2 - 1350+
000199	000	C02020+HE	=C02010+HE	14 3.8 -21-3.0 843.
000200	000	C02010+M3	=C02000+M3	64 3.4 -26-4.2 1130.
000201	000	C02010+HE	=C02000+HE	14 9.9 -22-3.0 843.
000202	000	C02001+N2(0)	=CO2000+N2(1)	12 8.3 -12 0.5
000203	000	C02010+H20	=C02000+H20	15 3.2 -13 45.5 0.333
000204	000	C02001+H20	=CO2O3O+H2O	11 4.0 -13
000205	000	N2(1) +H20	=N2(0) +H20	

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END ELT.

WADD,P DATA

aXQT SEAGULL

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PROGRAM SEAGULL SUPERSONIC INVISCID FLOW ANALYSIS NASA LANGLEY RESEARCH CENTER RELEASED 02/01/76 LAST MODIFICATION 02/25/77 TWO DIMENSIONAL FLOW CCCC 0000 C A S E *** OUTPUT AT EVERY 100 STEPS GAMMA= .1200+01 DISTANCE= .4300+03 TOLERANCE= .1000+02 SCALE= .3000+01 ICN= 1 IBN= 1 ISO= Ŭ ISY= Ũ ITY= Ũ IVI= Ũ -.3000+02 DEGREES BREAK IN THE UPPER WALL AT Z= .1000+01 -.3000+02 DEGREES BREAK IN THE LOWER WALL AT Z= .9980+00

INITIAL CONDITIONS

TEMPERATURE (DEG	. KELVIN)	2.0000000+03
VELOCITY (M/S)		4.5628436+03
PRESSURE (ATM)		2+0000000+00
DENSITY(G/CC)		2.2614246-04
MOLECULAR WT.		1.8554989+01
GAMMA		1.4520176+00
TOTAL ENTHALPY		2.1616025+12
ENTROPY		2.1521133+09
MOLE FRACTION	C02100	3.2200000-03
MOLE FRACTION	C02001	1.6200000-03
MOLE FRACTION	02000	8.7599999-03
MOLE FRACTION	C02110	1.0830000-02
MOLE FRACTION	C02010	1.0180000-02
MOLE FRACTION	C02020	8.7299999-03
MOLE FRACTION	C02030	3.9299999-03
MULE FRACTION	N2(0)	3.7432999-81
MULE FRACTION	N2(1)	7.0239999-02
MOLE FRACTION	02	5.909000-02
MULE FRACTION	H20	2.3630000-02
MOLE FRACTION	HE	4.2543999-01

CO2 G DL REA

= COZO30+ H2O

= N2(0) + H20

KF=A+LXP(B/RT++M)/T++N

•00

.00

NG) 1

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-139.0

.00

.33

			A	N	В	ж
C02110+ C02000	= C02100+ C02010	12	7.529+10	+.50	•0	.00
C02030+ C02000	= CO2100+ CO2010	1 2	1.084+09	50	•0	.00
C02030+ C02000	= CO2O2O+ CO2O10	12	1.867+11	50	•0	.00
CO2100+ CO2000	= C02010+ C02010	1 1	2.409+11	.00	.0	•00
· C02020+ C02000	= c02010 + c02010	12	8.432+11	50	•0	.00
C02001+ H1	= CO2110+ H1	6 4	6.626-04	-4.80	1484.0	.00
C02001+ M4	= C02110+ M4	6 4	1.144-07	-5.80	2436.0	.00
СО2001+ М1	= CO2O3O+ M1	64	4.879-07	-5.60	1484.0	.00
C02001+ M4	= CO2O3O+ M4	6 4	8.432-11	-6.60	2436.0	.00
C02110+ H2	= CO2O3O+ M2	6 Ż	2.590+07	-1.50	•0	.00
C02110+ M2	= CO2O2O+ M2	64	2.710-03	-4.20	-903.0	.00
C02110+ M2	= C02020+ H2	6 4	5.300+04	-2.50	-4410.0	.00
C02110+ M2	= C02100+ M2	64	5.180+00	-3.80	-549.0	•00
C02030+ H2	= CO2020+ M2	64	5.602+02	-3.30	-1230.0	.00
C02030+ M2	= CO2100+ M2	64	6.626+02	-3.00	-1060.0	•00
C02100+ H2	= CO2O2O+ M2	62	4.819+06	-1.50	•0	•00
C02100+ M3	= CO2D10+ M3	64	3.373+02	-3.30	-1480.0	.00
C02100+ HE	= C02010+ HE	1 4	1.084+03	-3.00	843.0	•00
CO2020+ M3	= CO2010+ M3	64	1.265+03	-3.20	-1350.0	.00
C02020+ HE	= CO2010+ HE	1 4	2.289+03	-3.00	843.0	•00
C02010+ M3	= CO2000+ M3	64	2.048-02	-4.20	1130.0	•00
C02010+ HE	= C02000+ HE	1 4	5.963+02	-3.00	843.0	.00
CO2001+ N2(0)	= CC2000 + N2(1)	12	4.999+12	•50	•0	.00
CO2010+ H20	= CO2000+ H20	15	1.927+11	•00	45.5	• 3 3

CTION

MODEL

CATALYTIC SPECIES BEING CONSIDERED

M1 = 1.00 CO2100, 1.00 CO2001, 1.00 CO2000, 1.00 CO2110, 1.00 CO2010, 1.00 CO2020, 1.00 CO2030, .CO N2(0), .OU N2(1), .OU O2 , 1.00 H20 , .OD HE ,

1 1 2.409+11

6.626+13

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- M2 = 1.00 CO2100, 1.00 CO2001, 1.00 CO2000, 1.00 CO2110, 1.00 CO2010, 1.00 CO2020, 1.00 CO2030, 1.00 N2(0), 1.00 N2(1), 1.00 O2 , 1.00 H20 , 1.50 HE ,
- H3 = 1.00 CO2100, 1.00 CO2001, 1.00 CO2000, 1.00 CO2110, 1.00 CO2010, 1.00 CO2020, 1.00 CO2030, .50 N2(D) , .50 N2(1) , .60 02 , 1.00 H20 , .00 HE ,
- M4 = .00 C02100, .00 C02001, .00 C02000, .00 C02110, .00 C02010, .00 C02020, .00 C02030, 2.00 N2(0), 2.00 N2(1), 2.40 02 , .00 H20 , 1.00 HE ,

K= D Z= .9950+00 DZ= .0000

C02001+ H20

N2(1) + H20

LOWER WALL FORCES FR= UPPER WALL FORCES FR=	.0000 .0000	F2= .0000 F2= .0000		.0000 .0000	VFR= VFR=	.0000 .0000	VFZ= VFZ=	•0000 •0000	VM= VM=	.0000 .0000
SUM OF FORCES	.0000	• 3000		.0000		.0000		•0000		.0000
FORCE ON COMPUTATIONAL	PLANE=	.0000 ARE	A= .00	000						
MOMENT ABOUT R AXIS=	.0000	MOMENT ABOU	T Z AXIS	s= .0000						

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I	N	R	P	U	W	T	S	M	ТНΕ	RHO	нтот
Ō	21	1100+03	.1000+01	.0000	482C+01	.1000+01	.0000	+4000+01	.0300	+1000+01	. 4047+01
ŭ	20	.1095+03	1000+01	.0000	.4820+01	.1000+01	.0000	•4000+01	•0000	1000+01	. 4347+01
ū	19	.1089+03	.1000+01	.0000	4820+01	+1000+01	•0000	•4000+U1	.0000	+1000+01	+4047+01
õ	18	.1084+03	.1000+01	.0000	·4820+01	.1000+01	.0000	•4000+C1	.0000	+1000+01	+4047+01
Ō	17	.1079+03	+1000+01	.0000	.4820+01	+1000+01	.0000	.4000+01	.0000	+1000+01	. 4047+01
ō	16	.1074+03	+1000+01	.0000	•482D+D1	1000+01	.0000	.4000+01	•0000	+1000+01	.4047+01
ũ	15	.1068+03	.1000+01	.0000	4820+01	+1000+01	.0000	•4000+01	•0000	+1000+01	.4047+01
ū	14	.1063+03	+1000+01	•0000	.4820+01	1000+01	•0000	•4000+01	•0000	+1000+01	+4047+01
ū	13	1058+03	1000+01	.0000	+4820+01	1000+01	.0000	+4000+01	.0000	1000+01	. 4047+01
a	12	.1053+03	.1000+01	.0000	·4820+01	1000+01	+0000	•4000+01	.0000	-1000+01	.4047+01
۵	11	.1047+03	.1000+01	.0000	+4620+01	1000+01	•0000	.4000+01	•0000	1000+01	.4047+01
G	10	+1042+03	.1000+01	.0000	4820+01	.1000+01	.0000	+4000+91	•0000	+1000+31	.4047+01
ō	9	.1037+03	+1000+01	.0000	+4820+01	+1000+01	.0000	.4300+01	•0000	-1000+01	.4047+01
Ð	8	+1032+03	+1000+01	.0000	4820+01	+1000+01	.0000	+4000+01	•0000	+1000+01	.4047+01
n	7	.1026+03	.1000+01	.0000	+4820+01	.1000+01	.0000	+4000+01	.0000	+1000+01	.4047+01
ñ	6	1021+03	.1000+01	.0000	.4820+01	.1000+01	.0000	.4000+01	.0000	+1000+01	.4047+01
ñ	5	.1016+03	.1000+01	.0000	+4820+01	1000+01	•0000	•4000+01	.0000	+1000+01	.4047+01
õ	4	.1011+03	.1000+01	.0000	.4820+01	.1000+01	.0000	+4000+01	.0000	1000+01	.4047+01
õ	ŝ	.1005+03	.1000+01	.0000	.4820+01	.1000+01	.0000	•4000+01	0000	+1000+01	.4047+01
ŏ	2	.1000+03	•1000+01	.0000	+4820+01	.1000+01	.0000	+4000+01	.0000	. 1000+01	.4047+01

MOLE FRACTIONS

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MOLE FRACTIONS

PT R(N)	N2(0)	N2(1)	02	H20	HE		
21110.00000	3.74330-01	7.0240ú-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
20109,47368	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
19108.94737	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
18108.42105	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
17107.89474	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
16107.36842	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
15106.84210	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
14106.31579	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
13105.78947	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
12105.26316	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
11104.73684	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
10104.21053	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
9103.68421	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
8103.15789	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
7102+63158	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
6102.10526	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
5101.57895	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
4101.05263	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00
3100.52632	3.74330-01	7.02400-02	5.90900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+90
2100.00000	3.74330-01	7.02400-02	5.98900-02	2.36300-02	4.25440-01	1.00000+00	1.45202+00

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4. Title and Subtitle				-	ort Date			
Incorporation of Coupled	l Nonequilibrium C	hemistry	Into a		il 1979			
Two-Dimensional Nozzle (Code (SEAGULL)			6. Perfo	orming Organization Code			
7. Author(s)				8. Perfo	orming Organization Report No.			
Alan W. Ratliff								
				10. Work	< Unit No.			
9. Performing Organization Name and Addr								
Lockheed Missiles & Spac			F	11. Cont	ract or Grant No.			
Huntsville Research & Er 4800 Bradford Blvd.	igineering Center			NAS	1-14754			
Huntsville, AL			ŀ	13 Type	of Report and Period Covered			
12. Sponsoring Agency Name and Address	· ····· ··········				tractor Report			
National Aeronautics and	Space Administra	tion	ŀ					
Washington, D.C. 20546	bpace manificula	LION		14. Spon	isoring Agency Code			
15. Supplementary Notes			<u></u>					
Langley Technical Monito	or: Manuel D. Sal	as						
Final Report								
16. Abstract								
A two-dimensional multiple shock nozzle code (SEAGULL) has been extended to include the effects of finite rate chemistry. The basic code that treats multiple shocks and contact surfaces was fully coupled with a generalized finite rate chemistry and vibrational energy exchange package. The modified code retains all of the original SEAGULL features plus the capability to treat chemical and vibra- tional nonequilibrium reactions. Any chemical and/or vibrational energy exchange mechanism can be handled as long as thermodynamic data and rate constants are available for all participating species.								
17. Key Words (Suggested by Author(s))			tion Statement					
Nozzle Flow		Unclas	sified -	Unlin	nited			
Finite Rate Chemistry Supersonic								
Shock Waves								
					Subject Category 34			
19. Security Classif. (of this report)	20. Security Classif. (of this	page)	21. No. of	Pages	22. Price*			
Unclassified	Unclassified		55	-	\$5.25			
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