

General Disclaimer

One or more of the Following Statements may affect this Document

- This document has been reproduced from the best copy furnished by the organizational source. It is being released in the interest of making available as much information as possible.
- This document may contain data, which exceeds the sheet parameters. It was furnished in this condition by the organizational source and is the best copy available.
- This document may contain tone-on-tone or color graphs, charts and/or pictures, which have been reproduced in black and white.
- This document is paginated as submitted by the original source.
- Portions of this document are not fully legible due to the historical nature of some of the material. However, it is the best reproduction available from the original submission.



(NASA-CR-161246) THERMAL ANALYSES OF A
MATERIALS PROCESSING FURNACE BEING DEVELOPED
FOR USE WITH HEAT PIPES Final Report
(Lockheed Missiles and Space Co.) 43 p
HC A03/MF A01

N79-25350

Unclas
23462

CSSL 20D G3/34



Lockheed

Missiles & Space Company, Inc.

HUNTSVILLE RESEARCH & ENGINEERING CENTER

Cummings Research Park
4800 Bradford Drive,
Huntsville, Alabama

THERMAL ANALYSIS
OF A MATERIALS PROCESSING
FURNACE BEING DEVELOPED
FOR USE WITH HEAT PIPES

FINAL REPORT

May 1979

Contract NAS8-33471

by

J. V. McAnally
S. J. Robertson

APPROVED:


S. V. Bourgeois, Supervisor
Industrial Environmental R&E Section


for J. S. Farrior
Resident Director

CONTENTS

Section		Page
	FOREWORD	ii
1	INTRODUCTION AND SUMMARY	1
2	THERMAL MODEL	2
3	COMPUTER PROGRAM	8
4	SAMPLE PROBLEM	12
5	CONCLUDING REMARKS	15
Appendixes		
A	Listing of Computer Program for Special Materials Processing Furnace Thermal Model	A-1
B	Listing of Input Data Cards for Sample Problem	B-1

1. INTRODUCTION AND SUMMARY

A special materials processing furnace is being developed by NASA-MSFC for use in forthcoming Spacelab missions to study the solidification under closely controlled conditions of various sample materials in the absence of gravity. The samples are to be rod shaped and will be subjected to both heating and cooling simultaneously with the heating and cooling distributed along the length of the rod by annular heat pipes concentric with the sample rod. The heat pipes will serve to maintain near-uniform temperature distributions along the hot and cold sections of the rod with a very steep temperature gradient in between.

The thermal model is based on a Lockheed-Huntsville developed Thermal Analyzer computer program. The model was developed to be very general to enable the simulation of variations in the furnace design and, hence, serve as an aid in finalizing the design.

The thermal model is described in Section 2, with a user's guide given in Section 3. Some preliminary results obtained in testing the model are given in Section 4.

2. THERMAL MODEL

The thermal model for the furnace was based on a number of preliminary design drawings and sketches provided by NASA-MSFC. A simplified schematic of the furnace is shown in Fig. 1. We concentrated our detailed thermal modeling efforts in the inner core region, including the sample, heat pipes, core tube (heating elements) and heat sink. Some of the outer region was not modeled exactly, since this would introduce considerable complexity into the model without significantly enhancing accuracy.

In modeling the furnace, we divided the overall model into three regions. The center section contains the core tube, hot heat pipe and surrounding insulation and housing and is labeled as Region 1. The end section opposite to the cold heat pipe and heat sink contains primarily structural stiffeners and insulation and is labeled Region 2. The end section containing the cold heat pipe and heat sink is labeled Region 3.

Heat is generated in the core tube and is distributed to the sample rod after being transferred through the muffle and hot heat pipe. Some heat is lost by conduction through the multilayer insulation to the external housing, where it is collected in a cooling jacket. The heat flowing through the sample into the cold heat pipe and heat sink is collected and removed through the heat sink.

A detailed schematic of the Region 1 nodal network is given in Fig. 2 with a table of conductors. In this configuration, the sample rod is fully extended into the heated region. The position of the sample can be altered to simulate translation from the heated region into the cooled region. We assumed that the heat pipe is an infinite conductor and, hence, is isothermal. The region within the muffle is partially pressurized by helium to enhance heat transfer by conduction. We modeled heat transfer in the radial direction

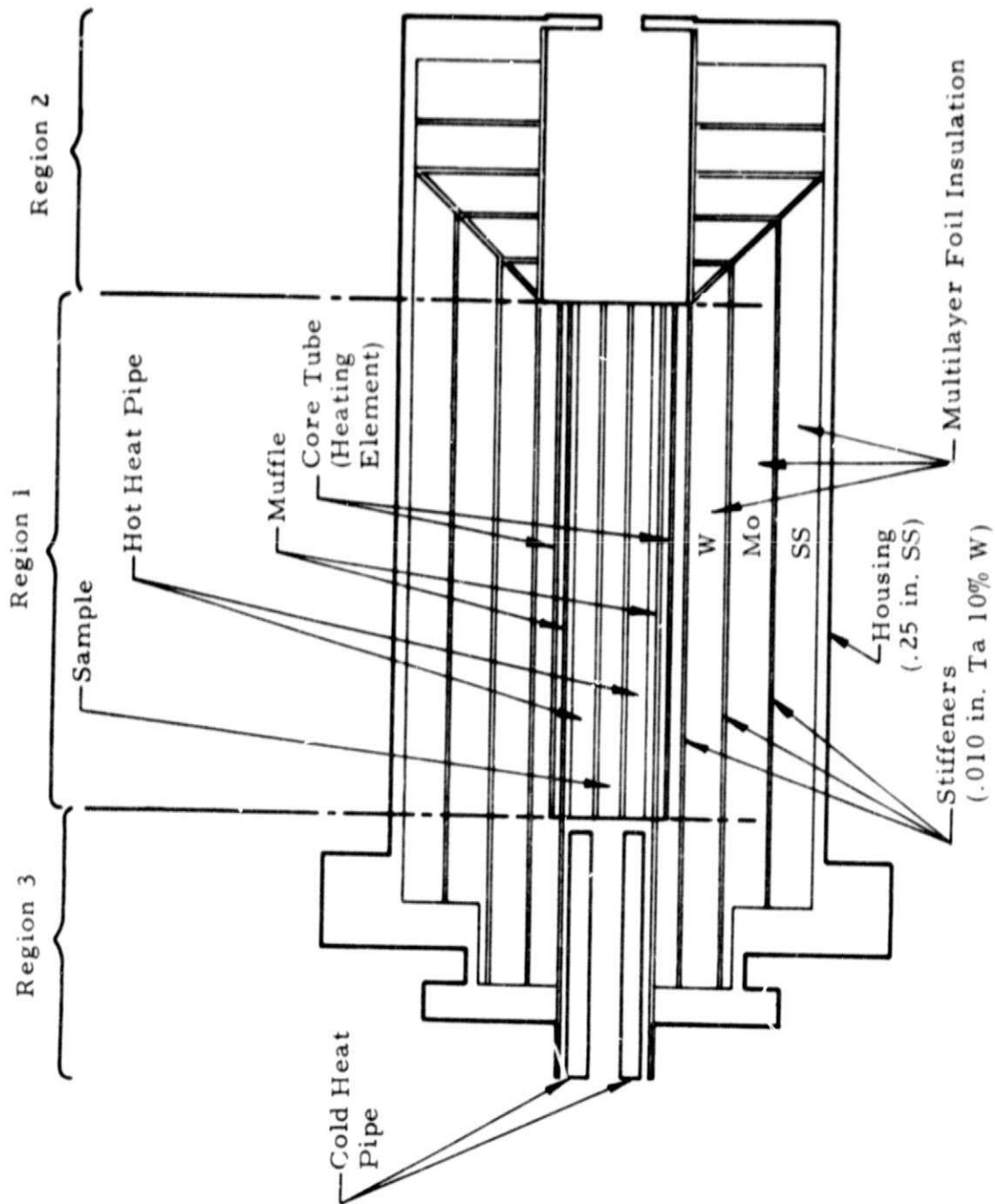


Fig. 1 - Schematic of Materials Processing Furnace

TABLE OF CONDUCTORS

Node Pair	Conductor	Type
1-2	1	Rad. & Cond.
2-3	2	Isothermal
3-4	3	Isothermal
4-5	4	Rad. & Cond.
...
9-10	9	Radiation
11-12	11	...
12-13	12	...
...
19-20	19	...
21-22	21	...
...
119-120	119	Conduction
1-11	120	Isothermal
3-13	122	...
5-15	124	...
...
11-21	130	...
13-23	132	...
...
...
101-111	220	...
103-113	222	...
105-115	224	...
...

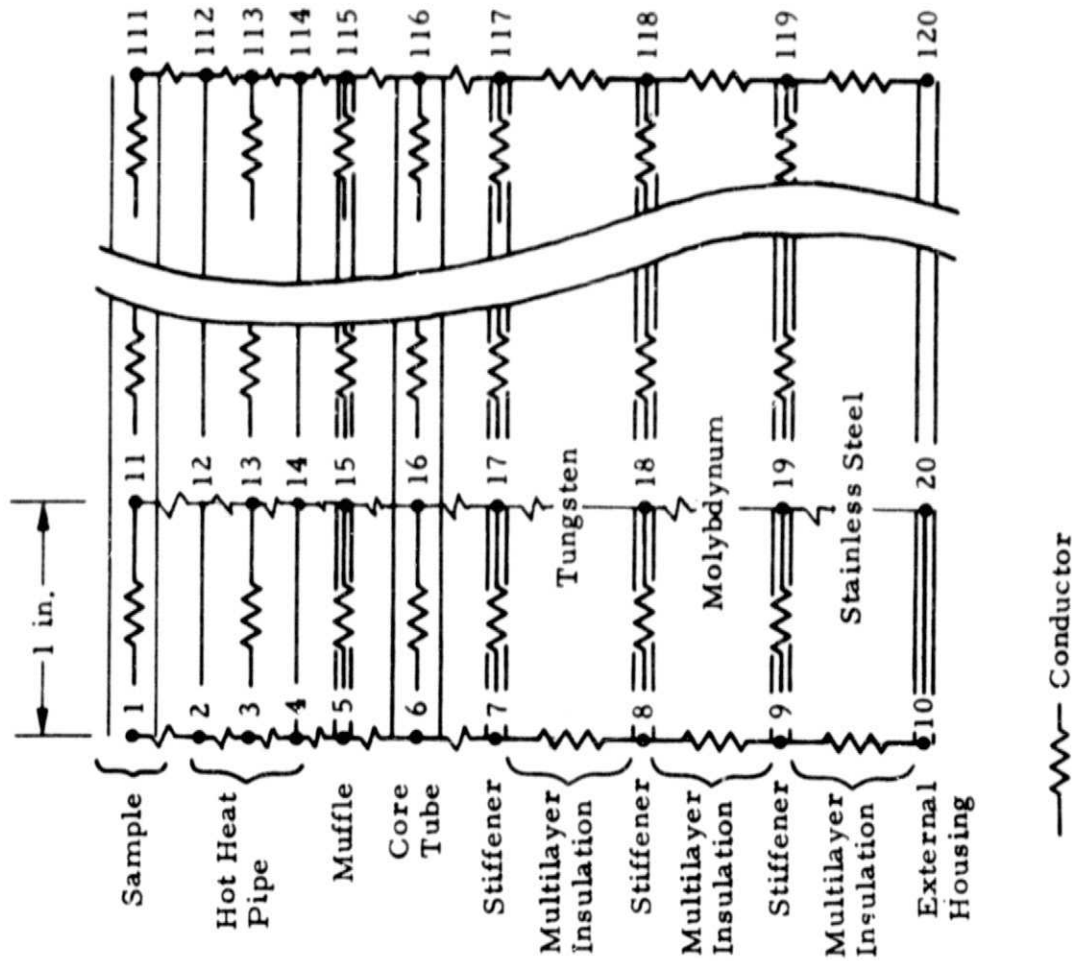


Fig. 2 - Schematic of Region 1 Nodal Network and Table of Conductors

within the muffle by a combination of conduction through the helium gas medium and radiation across the gaps separating the sample, heat pipe and muffle. Outside of the muffle, we modeled heat transfer in the radial direction by radiation only. Radiation through the multilayer foil is given by

$$q = \frac{\epsilon}{(2-\epsilon)n} (T_1^4 - T_2^4) \quad (1)$$

where q is the radiant heat flux, ϵ is the emissivity of the foil, n is the number of foil layers and T_1 and T_2 are the temperatures on the surfaces bounding the n foil layers.

Heat transfer in the axial direction was modeled by conduction along the structural elements.

A detailed schematic of the Region 2 nodal network and conductors is given in Fig. 3. The configuration of stiffeners and multilayer insulation which we modeled in Region 2 is a simplification of the actual design shown in Fig. 1. This simplification was made to facilitate the modeling of heat transfer in this region without unduly sacrificing accuracy. Heat transfer was modeled by conduction along the stiffeners and radiation through the multilayer insulation.

The Region 3 nodal network and conductors is shown in Fig. 4. The sample is thermally connected to the cold heat pipe only if it is translated into the cooled region. As in Region 1, heat transfer within the muffle is by a combination of conduction through the helium gas and radiation between structural members. Outside the muffle, radial heat transfer is by radiation only, and axial heat transfer is by conduction along structural members. The cold heat pipe, like the hot heat pipe, is assumed isothermal.

In Regions 1, 2 and 3, the external housing is maintained at a constant temperature by a cooling jacket. The furnace is assumed to be controlled by maintaining the four core tube sections at fixed temperatures. The cold heat pipe is maintained at a fixed temperature by cooling from some arbitrary heat sink.

TABLE OF CONDUCTORS

Node Pair	Conductor	Type
126-127	236	Radiation
127-128	237	
128-129	238	
129-130	239	
131-132	241	Conduction
132-133	242	
133-134	243	
134-135	244	
117-118	245	Radiation
118-119	246	
119-120	247	
117-135	246	
118-130	249	Radiation Radiation Conduction
119-130	250	
135-130	251	
130-121	252	
134-129	253	Radiation
129-122	254	
133-128	255	
128-123	256	
132-127	257	Radiation
127-124	258	
116-136	261	
115-136	262	
113-136	263	Radiation
111-136	264	

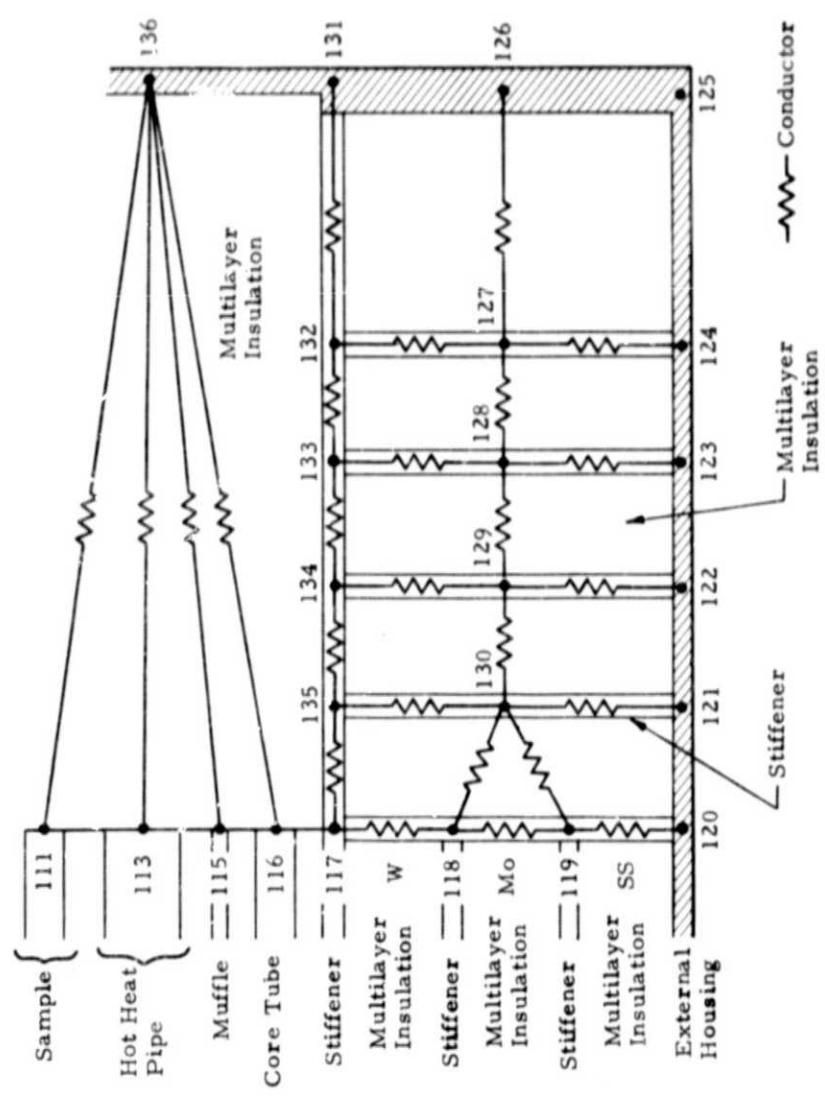


Fig. 3 - Schematic of Region 2 Nodal Network and Table of Conductors

TABLE OF CONDUCTORS

Node Pair	Conductor	Type
137-138	265	Conduction
138-139	266	
139-140	267	
140-141	268	
142-143	270	
143-144	271	
144-145	272	
145-146	273	
147-148	275	
148-149	276	
149-150	277	Cond. & Rad.
150-150	278	
152-153	280	
153-154	281	
154-155	282	
155-156	283	
158-159	286	
159-160	287	
137-142	290	
138-143	291	
139-144	292	
140-145	293	
141-146	294	
142-147	295	
143-148	296	
144-149	297	
145-150	298	
146-151	299	
148-153	301	
149-154	302	Conduction
150-155	303	
151-156	304	
153-157	306	
154-158	307	
155-159	308	
156-160	309	
158-161	310	
159-162	311	
160-163	312	
146-5	313	
151-7	314	
156-8	315	
160-9	316	Conduction

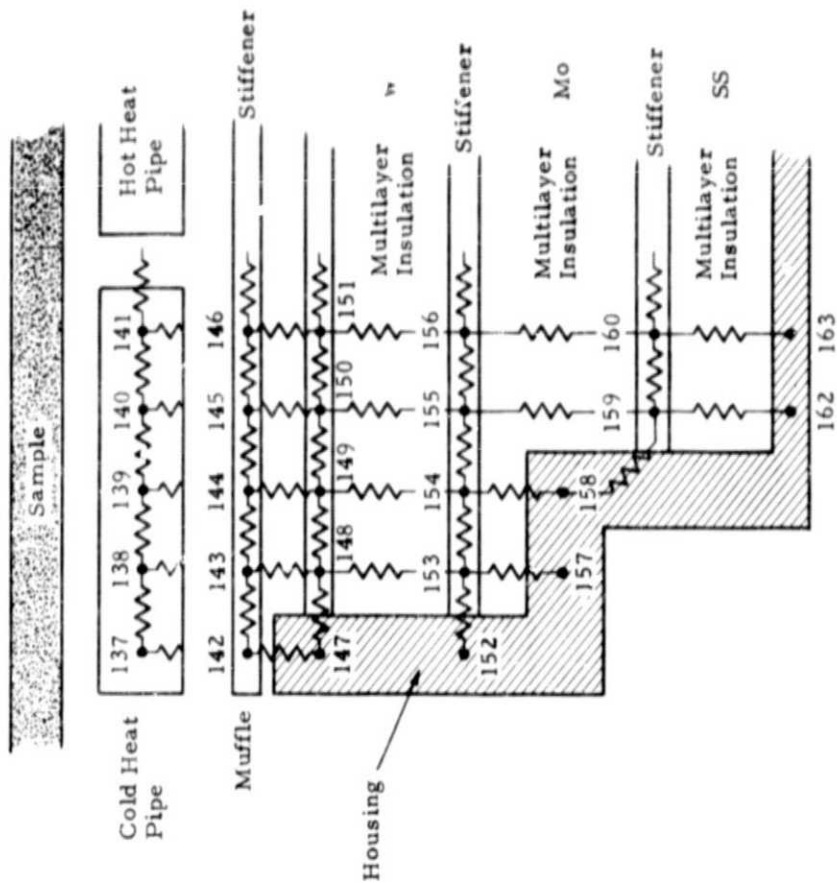


Figure 4 - Schematic of Region 3 Nodal Network and Table of Conductors

3. COMPUTER PROGRAM

A complete listing of the computer program for the thermal model is given in Appendix A. The program is coded in FORTRAN V language for the NASA-MSFC Univac 1108 computer. The program can be run for various core tube temperatures and sample translations with various heat sink and cooling jacket temperatures by varying the program inputs. The thermal model can be changed to simulate furnace design changes by modifying the program.

The main program contains the thermal analyzer coding, but calls subroutines for defining initial conditions, nodal networks, capacitances and conductances. Initial conditions for the temperatures are computed in subroutine CONI. The initial temperatures in Region 1 are computed based on fixed core tube and housing temperatures assuming radial conduction at equilibrium. The initial temperatures interior to the core tube in Region 1 are made equal to the core tube temperatures. The temperatures in Regions 2 and 3 are initialized at the cooling jacket, heat sink or some other temperature near a Region 1 temperature. The idea is to start the program calculations at initial temperatures as near as possible to the actual equilibrium temperatures. This approach to initializing temperatures reduces the required program run time.

The linking between nodes in the nodal network is defined in subroutine LINKUP, and conductances for each link are defined in subroutine COND. In COND, there are basically three kinds of conductances considered: (1) radiation, (2) solid conduction, and (3) helium conduction. The radiant conductance, CD, for a single gap is given by

$$CD = \frac{1}{1/\epsilon_1 + 1/\epsilon_2 - 1} \sigma(T_1^2 + T_2^2) (T_1 + T_2) A$$

where ϵ_1 and ϵ_2 are the emissivities and T_1 and T_2 are the temperatures of the two surfaces, σ is the Stephan-Boltzmann constant, and A is the area. The conductance for multilayer insulation is given by the same expression divided by the number of foil layers.

The conductance of solid conductors is given by

$$C_D = \frac{K}{\Delta X} A$$

where K is conductivity, ΔX is thickness and A is cross-sectional area. The K values are read into the program in table form in subroutine DATA as a function of temperature. The tabular data is interpolated in subroutine INTP. Provision is made for 16 tables in the program giving conductivity and specific heat for stainless steel, alumina, tungsten, molybdenum and tantalum alloy. In addition, tables are set aside for the sample material and the hot and cold heat pipes. The tables are described in the listing of COND in Appendix A.

Conductances are also calculated for helium conduction in the region contained within the muffle. These calculations are noted by comments in the program listing.

Since the program computes an equilibrium solution, the values of the capacitances are not significant to the final solution except when used to flag nodes where temperatures are held constant. When the value of the capacitance is -1, the node is flagged as a constant temperature node. This is done in subroutine CAP. Subroutine HTRT is provided for defining heat generation rates. In our case, however, we set all heat rates Q equal to zero and, instead, use constant temperature nodes at points of heat generation or loss.

The time step in the numerical solution is recomputed to ensure stability at each step in the solution. This is done in subroutine TSTP.

The program inputs and their formats are given in Table 1.

Table 1
COMPUTER PROGRAM INPUT DATA

Card	Column	Format	Variable	Description
1	1-10	E10.0	PRIN	Arbitrarily set equal to 100
1	11-15	I5	NPR1	Number of iterations between printouts
1	16-20	I5	NEXCT	Arbitrarily set equal to 100
1	21-30	E10.0	TSCON	Maximum stable time step divided by this number to assure stability
1	31-40	E10.0	TMAX	Program run time
1	41-50	E10.0	TCASE	Cooling jacket temperature, R
1	51-60	E10.0	TCOOL	Heat sink temperature, R
2	1-10	E10.0	TCORE(1)	Core tube temperature, R
2	11-20	E10.0	TCORE(2)	
2	21-30	E10.0	TCORE(3)	
2	31-40	E10.0	TCORE(4)	
3	1-5	I5	NTRANS	Sample translation distance, in.
4	1-5	I5	N	Table number
4	2-10	I5	J	Number of values in table
5	1-10	E10.0	A	Temperature, R
5	11-20	E10.0	B	Table value (conductivity or capacitance)
5	21-80	-	-	Comment

(Repeat card 5 J times)
(Repeat cards 4 and 5 sequence for each table. End with blank card).

The program output lists the program input data, the conductors and their nodal end points, the nodes and the connecting nodes and conductors, and the tables of material properties prior to entering the thermal analyzer loop. After entering the thermal analyzer loop, a listing is made of temperatures, heat rates and conductance values at specified intervals. After completion of a run, totals are made of heat transferred to the cooling jacket (QCASE) and to the cold heat pipe heat sink (QCOOL). The sum of these heat flow rates is the furnace requirement (POWER). The heat flow rates QCASE and QCOOL are printed out in units of kcal/sec, and the power level POWER is printed out in kilowatts.

4. SAMPLE PROBLEM

A sample case was run for a tungsten sample rod translated 5 in. into the cooled region. The four core tube temperatures were assumed to be 838 C (2000 R), with the external housing and cold heat pipe cooled to 27 C (540 R). The input cards for this case are listed in Appendix B. The values of PRIN and NEXCT are not significant to our model and were arbitrarily set equal to 100. NPR1 was set equal to 50 to print data every 50 iterations. TSCON was set equal to 2 to assure a stable solution. TMAX was set equal to 100, since we found by experience that equilibrium is reached by that time.

The calculated temperatures at equilibrium are plotted in Fig. 5 along a radius at the midpoint of the core tube. The calculated sample temperatures are plotted in Fig. 6 along the sample length. The total heat lost to the external housing cooling jacket QCASE was found to be 146 kcal/hr, and the heat flow to the cold heat pipe was found to be 255 kcal/hr. This corresponds to a total power level for the core tube of .47 kW.

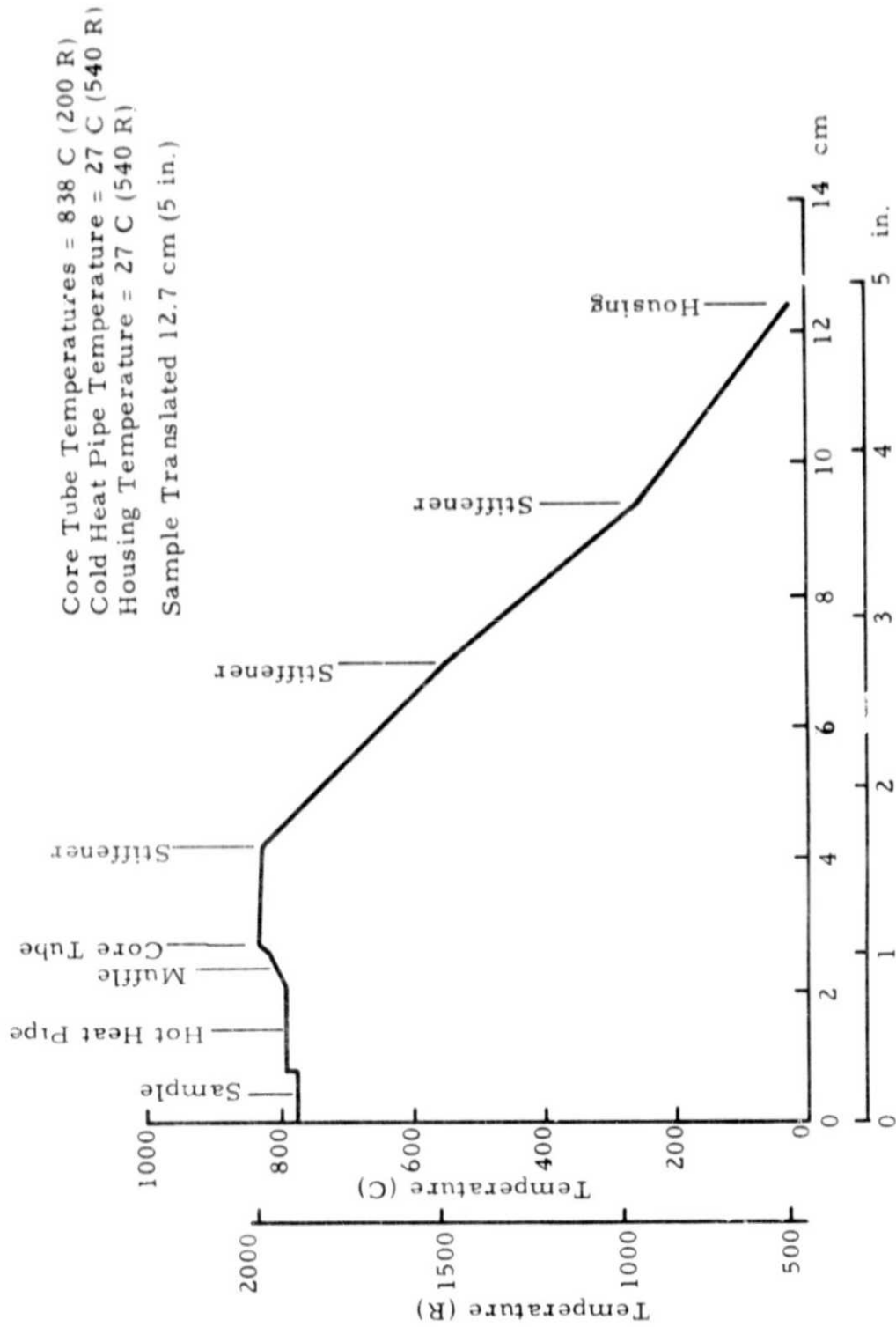


Fig. 5 - Temperature Distribution along Furnace Radius Bisecting Core Tube

Core Tube Temperatures = 838 C (200 R)
 Cold Heat Pipe Temperature = 27 C (540 R)
 Housing Temperature = 27 C (540 R)
 Sample Translated 12.7 cm (5 in.)

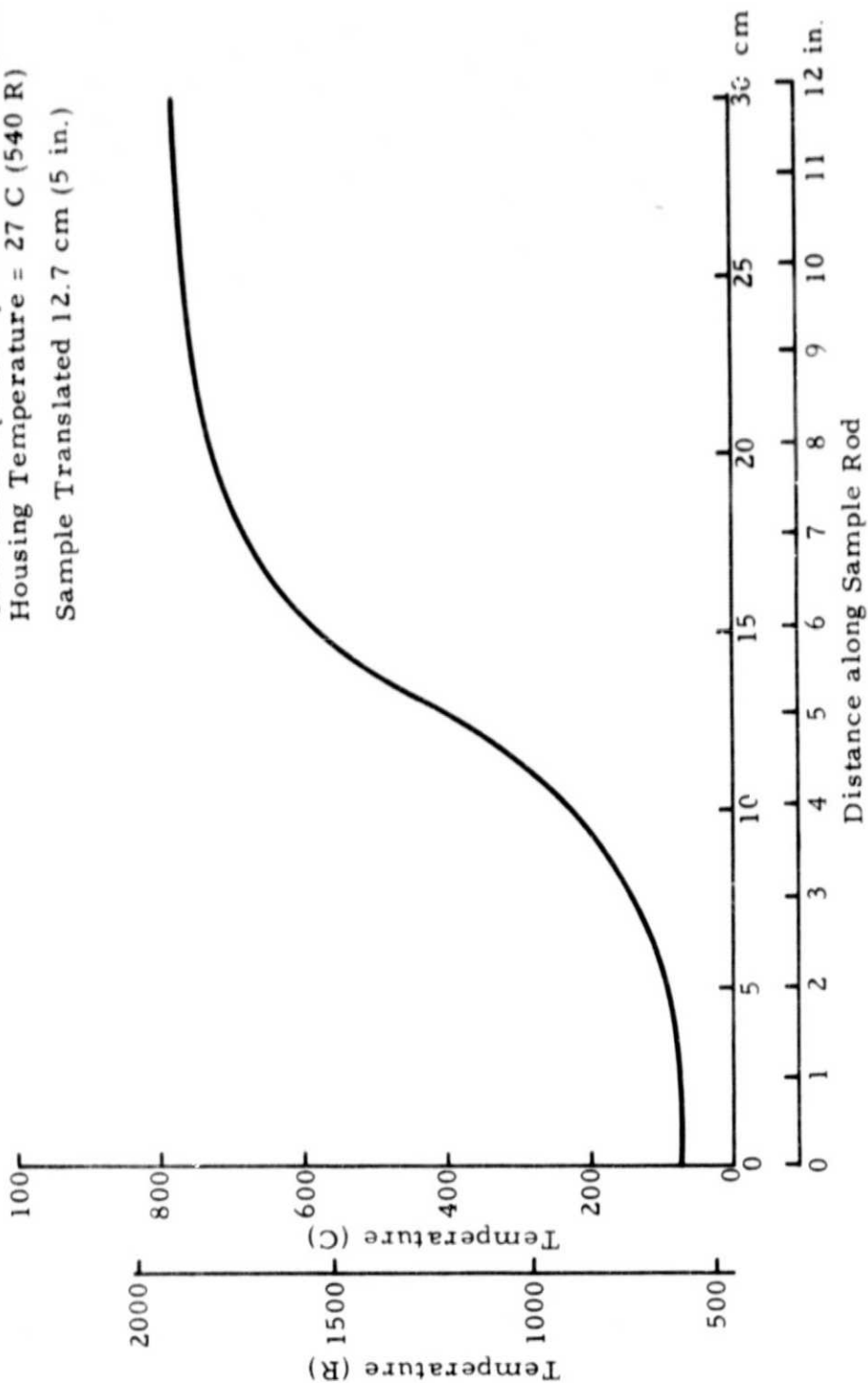


Fig. 6 - Temperature Distribution along Tungsten Sample Rod

5. CONCLUDING REMARKS

The thermal model developed during this effort accurately simulates heat transfer in the baseline furnace design, and is sufficiently general to model a broad range of design modifications. The model can be utilized to perform heat transfer analyses for various anticipated test conditions to determine required furnace power levels, cooling rates and sample temperature distributions.

It is recommended that experimental data be obtained on the conductance of the multilayer foil insulation. This updated data can be incorporated into the present thermal model with little difficulty.

Appendix A

LISTING OF COMPUTER PROGRAM FOR SPECIAL
MATERIALS PROCESSING FURNACE THERMAL MODEL

```

C*****
COMMON/TRANS/NTRANS
COMMON/XINIT/ICON1,CD67,CD78,CD89,CD910
C          T H E R M A L   A N A L Y S E R          B Y   J O H N   P O N D
C*****
DIMENSION T(200),C(200),CD(500),G(200),TO(200)
DIMENSION L(500,2),M(200,15)
DIMENSION TCORE(4)
701 FORMAT(2I5,E10.0)
702 FORMAT (4I5)
703 FORMAT(E10.0,2I5,4E10.0)
704 FORMAT(4E10.0)
705 FORMAT(1H1,'PRIN=',F10.0,2X,'NPR1=',I5,2X,'NEXCT=',I5,2X,'TSCON=',
  & F10.0,2X,'TMAX=-',F10.0)
706 FORMAT(1H0,'CASE=',F10.0,2X,'TCOOL=',F10.0,2X,4('TCORE(',I1,')=',
  & F10.0,2X,))
707 FORMAT(1H0,'NTRANS=',I2)
  1          15,3X,6HNIT = ,I5)
  & F10.0,2X,))
707 FORMAT(1H0,'NTRANS=',I2)
  1          15,3X,6HNIT = ,I5)
711 FORMAT(1H , (2HT(,I5,2H)=,2X,E10.4,3X,
  1          2HT(,I5,2H)=,2X,E10.4,3X,
  2          2HT(,I5,2H)=,2X,E10.4,3X,
  3          2HT(,I5,2H)=,2X,E10.4,3X,
  4          2HT(,I5,2H)=,2X,E10.4,3X))
712 FORMAT(1H , (2HC(,I5,2H)=,2X,F8.3,3X,
  1          2HC(,I5,2H)=,2X,F8.3,3X,
  2          2HC(,I5,2H)=,2X,F8.3,3X,
  3          2HC(,I5,2H)=,2X,F8.3,3X,
  4          2HC(,I5,2H)=,2X,F8.3,3X))
713 FORMAT(1H , (2HK(,I5,2H)=,2X,E10.4,3X,
  1          2HK(,I5,2H)=,2X,E10.4,3X,
  2          2HK(,I5,2H)=,2X,E10.4,3X,
  3          2HK(,I5,2H)=,2X,E10.4,3X,
  4          2HK(,I5,2H)=,2X,E10.4,3X))
714 FORMAT(1H , (2HQ(,I5,2H)=,2X,E10.4,3X,
  1          2HQ(,I5,2H)=,2X,E10.4,3X,
  2          2HQ(,I5,2H)=,2X,E10.4,3X,
  3          2HQ(,I5,2H)=,2X,E10.4,3X,
  4          2HQ(,I5,2H)=,2X,E10.4,3X))
715 FORMAT(1H0,3I5)
716 FORMAT(1H0,2I5/(1H ,20I5))
717 FORMAT(1H1,24HCONDUCTOR NODE-1 NODE-2)
718 FORMAT(1H1,25HNODE NUMBER OF RESISTORS,/)
  1          1H ,10HNODE--CON.,5X,10HNODE--CON.,5X,
  2          1H ,10HNODE--CON.,5X,10HNODE--CON.,5X,
  3          1H ,10HNODE--CON.,5X,10HNODE--CON.,5X,
  4          1H ,10HNODE--CON.,5X,10HNODE--CON.,5X)

```

```

720 FORMAT(/:IX,18HMAX TEMP CHANGE = ,F10.5,3X,9HAT NODE  ,15,
1      26H5UM OF ABS TEMP CHANGES = ,E15.6)
730 FORMAT(1H ,(5HQDUT(.15,2H)=,2X,E12.4,3X,
1      5HQDUT(.15,2H)=,2X,E12.4,3X,
2      5HQDUT(.15,2H)=,2X,E12.4,3X,
3      5HQDUT(.15,2H)=,2X,E12.4,3X))
2000 CONTINUE
      READ(5,703) PRIN,NPR1,NEXCT,TSCON,TMAX,TCASE,TCOOL

      READ(5,704)(TCORE(I),I=1,4)
      READ(5,702)NTRANS
      WRITE(6,705)PRIN,NPR1,NEXCT,TSCON,TMAX
      WRITE(6,706)TCASE,TCOOL,(I,TCORE(I),I=1,4)
      WRITE(6,707)NTRANS

C
C*****
      IF (PRIN.EQ.0.0) GO TO 4000
      NNDS=163
      NCDS=347
      KKFLG=100
      CALL CONI(T,TCORE,TCASE,TCOOL,CD,L)
      CALL LINKUP(L)
C*****
C      SORT OUT CONDUCTORS AND ADJACENT NODES
C
C*****
      DO 606 IN = 1,NNDS
      NR = 0
      DO 607 IR = 1,NCDS
      IF(L(IR,1).NE.IN) GO TO 60B
      NR = NR + 1
      I1 = NR*2
      I2 = NR*2+1
      M(IN,I1) = L(IR,2)
      M(IN,I2) = IR
      GO TO 609
60B CONTINUE
      IF(L(IR,2).NE.IN) GO TO 609
      NR = NR + 1
      I1 = NR*2
      I2 = NR*2+1
      M(IN,I1) = L(IR,1)
      M(IN,I2) = IR
609 CONTINUE
607 CONTINUE
      M(IN,1) = NR
606 CONTINUE
C*****
C
C      PRINT OUT INITIAL INFORMATION
C

```

ORIGINAL PAGE IS
OF POOR QUALITY

```

C*****
C   WRITE(6,717)
C   DO 502 I = 1,NCDS
C   WRITE(6,715) I,L(1,1),L(1,2)
C502  CONTINUE
      WRITE(6,718)
      DO 500 I=1,NNDS
      IN=M(1,1)*2+1
      WRITE(6,716) I,(M (I,J),J=1,IN)
500  CONTINUE
      TIM = 0.0
      NIT = 0
      IPT=1
      NPR = 0
      DT = 0.0
      NCTRL = 0

      MAXTND = 0
      TMAXCG = 0.0
      TSMCG = 0.0

C
C   READ IN TABULAR DATA
C
C   CALL DATA
C*****
C                               START LOOP
      DO 1012 I=1,NNDS
      C(I)=2.0
1012  CONTINUE
1000  CONTINUE
      CALL CAP(C,T)
      CALL COND(CD,L,T)

C
C*****
C
C   COMPUTE HEAT RATES
      CALL HTRT(T,TIM,DT,NNDS,Q)
152  CONTINUE

C
C*****
C
C   PRINT OUT RESULTS
C*****
      IF(NPR1*(NPR/NPR1).NE.NPR) GO TO 160
      WRITE(6,710) TIM,DT,NCTRL,NIT
      WRITE(6,720) TMAXCG,MAXTND,TSMCG
      WRITE(6,711) (N,T(N),N=1,NNDS)
      WRITE(6,714) (N,Q(N),N=1,NNDS)
      WRITE(6,713) (N,CD(N),N=1,NCDS)

```

```

160 CONTINUE
    NPR = NPR + 1
150 CONTINUE
C
C*****
C
C    COMPUTE TIME STEP
    CALL TSTEP(C,CD,NCTRL,M,NNDS,NPR,PRIN,TIM,IPT,TSCON,DT)
C
C*****
    MAXTND = 0.0
    TMAXCG = 0.0
    TSMCG = 0.0
    TIM = TIM + DT
    NIT=NIT+1
    DO 800 I = 1,NNDS
        TCG = ABS(T(I)-TJ(I))
        IF(TCG.LT.TMAXCG) GO TO 945
        MAXTND = I
        TMAXCG = TCG
    945 CONTINUE
        TSMCG=TSMCG+TCG
        TO(I) = T(I)
    800 CONTINUE
C*****
C
C    COMPUTE NEW TEMPERATURES
C
C*****
    DO 900 I = 1,NNDS
        IF(C(I))903,902,901
C
C    TRANSIENT CALCULATION
C
    901 CONTINUE
        NADJ = M(I,1)
        F1 = 0
        F2 = 0
        DO 803 J = 1,NADJ
            NR1 = J*2
            NR2 = J*2+1
            JN = M(I,NR1)
            JC = M(I,NR2)
            F1 = F1 + TO(JN)*CD(JC)
            F2 = F2 + TO(I)*CD(JC)
        803 CONTINUE
        T(I) = TO(I)+(F1-F2+Q(I))*DT/C(I)
        GO TO 900

```



```

C
C   STEADY STATE CALCULATION
C
902 CONTINUE
   NADJ = M(1,1)
   F1 = 0
   F2 = 0
   DO 804 J = 1,NADJ
     NR1 = J*2
     NR2 = J*2+1
     JN = M(1,NR1)
     JC = M(1,NR2)
     F1=F1+T0(JN)*CD(JC)
     F2 = F2 + CD(JC)
804 CONTINUE
   T(1)=(F1+Q(1))/F2
903 CONTINUE
900 CONTINUE

C
C   HOT HEAT PIPE TEMPERATURE ASSUMING INFINITE CONDUCTOR
C
   SUM1=0.
   SUM2=0.
   SUM3=0.
   SUM4=0.
   IMAX=12-NTRANS
   DO 290 I=1,IMAX
     K=1+10*(I-1)
     SUM1=SUM1+T(K)*CD(K)
     SUM2=SUM2+CD(K)
290 CONTINUE
     DO 291 I=1,12
       K=4+10*(I-1)
       SUM3=SUM3+T(K+1)*CD(K)
       SUM4=SUM4+CD(K)
291 CONTINUE

     SUM3=SUM3+T(141)*CD(347)
     SUM4=SUM4+CD(347)
     T(3)=(SUM1+SUM3)/(SUM2+SUM4)
     DO 300 I=2,112,10
       T(1)=T(3)
       T(I+1)=T(3)
       T(I+2)=T(3)
300 CONTINUE
     IF(TIM.LE.TMAX) GO TO 1000
     NPTS=NPK
C

```

```

C      HEAT RATES
C
      QCASE=0.
      DO 320 I=9,119,10
      QCASE=QCASE+CD(I)*(T(I)-TCASE)
320   CONTINUE
      QCASE=QCASE+CD(247)*(T(119)-T(120))
      QCASE=QCASE+CD(252)*(T(130)-T(121))
      QCASE=QCASE+CD(254)*(T(129)-T(122))
      QCASE=QCASE+CD(256)*(T(128)-T(123))
      QCASE=QCASE+CD(258)*(T(127)-T(124))
      QCASE=QCASE+CD(236)*(T(127)-T(126))
      QCASE=QCASE+CD(241)*(T(132)-T(131))
      QCASE=QCASE+CD(311)*(T(159)-T(162))
      QCASE=QCASE+CD(312)*(T(160)-T(163))
      QCASE=QCASE+CD(286)*(T(159)-T(158))
      QCASE=QCASE+CD(307)*(T(154)-T(158))
      QCASE=QCASE+CD(306)*(T(153)-T(157))
      QCASE=QCASE+CD(280)*(T(153)-T(152))
      QCASE=QCASE+CD(275)*(T(148)-T(147))
      QCASE=QCASE+CD(295)*(T(142)-T(147))
      QCOOL=0.
      DO 330 I=1,5
      QCOOL=QCOOL+(CD(289+I)+CD(341+I))*(T(141+I)-TCOOL)
330   CONTINUE
      QCOOL=QCOOL+CD(347)*(T(3)-T(141))
      IF(NTRANS.EQ.0) GO TO 340
      DO 335 I=1,NTRANS
      K1=1+10*(11+I-NTRANS)
      K2=329-NTRANS+I
      J=1+10*(I-1)
      QCOOL=QCOOL+(CD(K1)+CD(K2))*(T(J)-TCOOL)
335   CONTINUE
340   CONTINUE
      QCASE=QCASE*(.252/3600.)
      QCOOL=QCOOL*(.252/3600.)
      POWER=(QCASE+QCOOL)*4.186
      WRITE(6,350)QCASE,QCOOL,POWER
350   FORMAT(1H0,'QCASE=',E10.3,' KCAL/SEC',2X,'QCOOL=',E10.3,
$ ' KCAL/SEC',
$ 2X,'POWER=',E10.3,' KW')
      GO TO 2000
4000  STOP
      END

```

```

SUBROUTINE CON1(T,TCORE,TCASE,TCOOL,CD,L)
COMMON/TRANS/NTRANS
COMMON/XINIT/ICON1,CD67,CD78,CD89,CD910
DIMENSION T(200),TCORE(4),CD(500),L(500,2)

C
C C
C     INITIAL TEMPERATURES, REGION 1.

    ICON1=0
    CALL COND(CD,L,T)
    ICON1=1
    DO 10 I=6,116,10
    J=(I-6)/30+1
    T(I)=TCORE(J)
    T(I+4)=TCASE
    QCONST=(T(I)**4-T(I+1)**4)/(1./CD67+1./CD78+1./CD89+1./CD910)
    T(I+1)=(T(I)**4-QCONST/CD67)**.25
    T(I+2)=(T(I+1)**4-QCONST/CD78)**.25
    T(I+3)=(T(I+2)**4-QCONST/CD89)**.25
    T(I-1)=T(I)
    T(I-2)=T(I)
    T(I-3)=T(I)
    T(I-4)=T(I)
    K=I-5+NTRANS
    IF(K.GT.111)GO TO 10
    T(K)=T(I)
10 CONTINUE

C
C C
C     INITIAL TEMPERATURES, REGION 2.

    DO 20 I=121,126
    T(I)=TCASE
20 CONTINUE
    T(131)=TCASE
    T(136)=TCASE
    T(135)=T(118)
    T(134)=T(119)
    T(133)=TCASE
    T(132)=TCASE
    T(127)=TCASE
    T(128)=TCASE
    T(129)=0.5*(T(134)+TCASE)
    T(130)=0.5*(T(135)+TCASE)

C
C C
C     INITIAL TEMPERATURES, REGION 3.

    DO 30 I=137,141
    T(I)=TCOOL
30 CONTINUE
    DO 40 I=142,163
    T(I)=TCASE

```

```
40 CONTINUE
   IF(NTRANS.EQ.0) GO TO 50
   DO 45 I=1,NTRANS
45  T(I)=TCOOL
50  CONTINUE
   RETURN
   END
```

```
      SUBROUTINE HTRT(T,TIM,DT,NNDS,Q)
      DIMENSION T(200),Q(200)
      DO 1 I=1,NNDS
      Q(I)=0.
1  CONTINUE
   RETURN
   END
```

```
      SUBROUTINE CAP(C,T)
      DIMENSION T(200),C(200)
      DO 5 I=6,116,10
      C(I)=-1.0
5  CONTINUE
      DO 2 I=137,141
      C(I)=-1.00
2  CONTINUE
      DO 3 I=10,120,10
      C(I)=-1.00
3  CONTINUE
      DO 4 I=121,126
      C(I)=-1.0
4  CONTINUE
      C(131)=-1.00
      C(136)=-1.00
      C(162)=-1.00
      C(163)=-1.00
      C(161)=-1.00
      C(158)=-1.00
      C(157)=-1.00
      C(152)=-1.00
      C(142)=-1.0
      C(147)=-1.0
      RETURN
      END
```

```

SUBROUTINE LINKUP(L)
COMMON/TRANS/NTRANS
DIMENSION L(500,2),LL(500,2)
DATA(LL(J,1),J=245,264)/
1  117,118,119,117,118,119,135,130,134,129,
2  133,128,132,127,126,131,116,115,113,111/
DATA(LL(J,2),J=245,264)/
1  118,119,120,135,130,130,130,121,129,122,
2  128,123,127,124,131,136,136,136,136,136/
DATA(LL(J,1),J=305,317)/
1  152,153,154,155,156,158,159,160,146,151,156,160,163/
DATA(LL(J,2),J=305,317)/
1  157,157,158,159,160,161,162,163,5,7,8,9,10/

C
C
C   CONDUCTOR HOOK-UPS FOR REGION 1

DO 100 J=1,119
L(J,1)=J
L(J,2)=J+1
100 CONTINUE
DO 200 J=120,229
JJ=J-119
L(J,1)=JJ
L(J,2)=JJ+10
200 CONTINUE
C   RADIATION BETWEEN SAMPLE AND HOT HEAT PIPE
IMAX=12-NTRANS
DO 110 I=1,IMAX
K=1+10*(I-1)
L(K,1)=K+NTRANS*10
L(K,2)=K+1
C   HE CONDUCTION BETWEEN SAMPLE AND HOT HEAT PIPE
J=317+I
L(J,1)=K+NTRANS*10
L(J,2)=K+1
110 CONTINUE
DO 111 I=1,12
C   RADIATION BETWEEN HOT HEAT PIPE AND MUFFLE
K=4+10*(I-1)
L(K,1)=K
L(K,2)=K+1
C   HE CONDUCTION BETWEEN HOT HEAT PIPE AND MUFFLE
J=329+I
L(J,1)=K
L(J,2)=K+1
111 CONTINUE

C
C
C   CONDUCTOR HOOK-UPS FOR REGION 2

```

```

DO 300 J=230,244
  JJ=J-110
  L(J,1)=JJ
  L(J,2)=JJ+1
300 CONTINUE
C
C   CONDUCTOR HOOK-UPS FOR REGION 3
C
DO 400 J=265,290
  JJ=J-128
  L(J,1)=JJ
  L(J,2)=JJ+1
400 CONTINUE
DO 500 J=290,304
  JJ=J-153
  L(J,1)=JJ
  L(J,2)=JJ+5
500 CONTINUE
DO 700 J=245,264
  L(J,1)=LL(J,1)
  L(J,2)=LL(J,2)
700 CONTINUE
DO 800 J=305,317
  L(J,1)=LL(J,1)
  L(J,2)=LL(J,2)
800 CONTINUE
  IF(NTRANS.EQ.0)GO TO 850
DO 845 I=1,NTRANS
C   RADIATION BETWEEN SAMPLE AND COLD HEAT PIPE
  K=1+10*(11+I-NTRANS)
  L(K,1)=(I-1)*10+1
  L(K,2)=141-NTRANS+I
C   HE CONDUCTION BETWEEN SAMPLE AND COLD HEAT PIPE
  K=329-NTRANS+1
  L(K,1)=(I-1)*10+1
  L(K,2)=141-NTRANS+I
  K=343+I
  L(K,1)=I
  L(K,2)=141-NTRANS+I
845 CONTINUE
850 CONTINUE
DO 846 I=1,5
C   HE CONDUCTION BETWEEN COLD HEAT PIPE AND MUFFLE
  J=341+I
  L(J,1)=136+I
  L(J,2)=141+I
846 CONTINUE
C   CONDUCTION BETWEEN HOT AND COLD HEAT PIPES
  L(347,1)=3
  L(347,2)=141
  WRITE(6,600)(J,L(J,1),L(J,2),J=1,347)
600 FORMAT(1X,3I5,5X,3I5,5X,3I5,5X,3I5,5X,3I5,5X,3I5)
  RETURN
  END

```

```

SUBROUTINE COND(CD,L,T)
COMMON/TRANS/NTRANS
COMMON/XI,IT/ICUN1,CD67,CD78,CD89,CD910
DIMENSION CD(500),L(500,2),T(200)
DIMENSION R(10),XK(500,16),RR(10)
DATA E1,E2,E4,E5,E6,E7,E8,E9,EW,LM,ES / .400,.2,.2,.328,.400,.328,
#.328,.328,.4,.2,.500/
DATA(RR(1),I=1,10)/.315,.330,.580,.830,1.007,1.170,1.650,2.750,
#.3,200,4,900/
DATA THTR,TSIF1,TWAL1,TMUF1,DELXX/.115,.010,.250,.015,1.0/
    
```

ASSUMPTIONS USED IN CALCULATING CONDUCTOR VALUES

REGION 1

- (1) .030 INCH CLEARANCE BETWEEN SAMPLE AND HEAT PIPE
- (2) .340 INCH CLEARANCE BETWEEN HEAT PIPE AND MUFFLE
- (3) .020 INCH CLEARANCE BETWEEN MUFFLE AND HEATING ELEMENT
- (4) .480 INCH CLEARANCE BETWEEN HEATING ELEMENT AND FIRST STIFFENER
- (5) .010 STIFFENER THICKNESS

NSW,NSM,NSS ARE THE NUMBER OF LAYERS OF TUNGSTEN, MOLY AND ST STEEL

SIGMA=1.714E-09

NSW=30

NSM=30

NSS=30

THE UNITS ON R,DELX,THTR,TSIF,TWAL,TMUF ARE FT. THE VALUES ARE READ IN INCHES AND CONVERTED TO FT.

DELX IS AXIAL DISTANCE STEP SIZE

TMUF IS THE MUFFLE THICKNESS

THTR IS THE HEATER THICKNESS

TSTF IS THE STIFFENER THICKNESS

TABLE NO.	MATERIAL	PROPERTY
1	ST. STEEL	CP
2	ST. STEEL	K
3	ALUMINA	CP
4	ALUMINA	K
5	TUNGSTEN	CP
6	TUNGSTEN	K
7	MOLY	CP
8	MOLY	K
9	TA-U/W	CP
10	TA-U/W	K
11	SAMPLE	CP
12	SAMPLE	K
13	HEAT PIPE	CP
14	HEAT PIPE	K
15	COLD H.P.	CP
16	COLD H.P.	K

C

```

XK,4E=0.063
ECHP=.400
DO 10 I=1,10

```

```

10 R(1)=RK(1)/12.
DELX=DELXX/12.
THTR=THTR1/12.
TSTF=TSTF1/12.
TWAL=TWAL1/12.
TMUF=TMUF1/12.
PI=3.1415927
A2=2.*PI*R(2)*DELX
A3=2.*PI*R(3)*DELX
A4=2.*PI*R(4)*DELX
A5=2.*PI*R(5)*DELX
A6=2.*PI*R(6)*DELX
A7=2.*PI*R(7)*DELX
A8=2.*PI*R(8)*DELX
A9=2.*PI*R(9)*DELX
A10=2.*PI*R(10)*DELX
R125 = 5.*DELX
R126 = 3.5*DELX
R131 = R(7)
AM235 = PI*(R125+R126)*DELX
AM259 = PI*(R126+R131)*DELX
F21=R(1)/R(2)
F34=R(4)/R(5)
F65=R(5)/R(6)
F111=1.0
F115=1.0
F118=1.0
F127=1.0
F116=1.0
F113=1.0
F119=1.0
F128=1.0

```

C
C
C
C

```

CALL INTERPOLATION ROUTINE WITH PREVIOUS TEMPERATURES TO DETERMINE
NEW CONDUCTIVITY

```

```

DO 700 J=2,16,2
DO 700 I=1,317
N1=L(I,1)
N2=L(I,2)
TAVG=(T(N1)+T(N2))/2.0
CALL INTP(TAVG,J,XKK)
XK(I,J)=XKK

```


700 CONTINUE

```

CD21 = (1./(1./E2+1./E1-1.))*SIGMA*A2*F21
CD32 = XK(1,14)*PI*(R(3)+R(2))/(R(3)-R(2))*DELX
CD43 = XK(1,14) * PI*(R(4)+R(3))/(R(4)-R(3))*DELX
CD54=(1./(1./E5+1./E4-1.))*SIGMA*A5*F54
CD65 = (1./(1./E6+1./E5-1.))*SIGMA*A6*F65
CD67 = (1./(1./E6+1./E7-1.))*SIGMA*A6
CD78 = (EW/(2.-EW))/(NSW+(2.*EW)/(2.-EW))*(1./E7+1./EW-1.)*
* SIGMA*A7
CD89 = (EM/(2.-EM))/(NSM+(2.*EM)/(2.-EM))*(1./E8+1./EM-1.)*
* SIGMA*A8
CD910 = (ES/(2.-ES))/(NSS+(2.*ES)/(2.-ES))*(1./E9+1./ES-1.)*
* SIGMA*A9
IF(ICON1.EQ.0)RETURN
A237=PI*(R(10)**2-R(7)**2)

```

```

CD237=CD78/A7*A237*F128
CD261 = (1./(1./E6+1./E5))*SIGMA*PI
CD290 = (1./(1./E5+1./E4HP))*SIGMA*A5*F54
CD296 = (1./(1./E5+1./E7))*SIGMA*A7*R(5)/R(7)
CD301 = CD78*A8/A7*R(7)/R(8)
CD306 = CD89*A9/A8*R(8)/R(9)
CD311 = CD910*A10/A9*R(9)/R(10)
DO 100 I=1,11,10
K=1+10*(I-1+NTRANS)
IF(K.LE.111)

```

```

*CD(I)=CD21*(T(K)**2+T(3)**2)*(T(K)+T(3))
CD(I+1)=CD32*XK(I+1,14)/XK(1,14)
CD(I+2)=CD43*XK(I+2,14)/XK(1,14)
CD(I+3)=CD54*(T(I+3)**2+T(I+4)**2)*(T(I+3)+T(I+4))
CD(I+4)=CD65*(T(I+4)**2+T(I+5)**2)*(T(I+4)+T(I+5))
CD(I+5)=CD67*(T(I+5)**2+T(I+6)**2)*(T(I+5)+T(I+6))
CD(I+6)=CD78*(T(I+6)**2+T(I+7)**2)*(T(I+6)+T(I+7))
CD(I+7)=CD89*(T(I+7)**2+T(I+8)**2)*(T(I+7)+T(I+8))
CD(I+8)=CD910*(T(I+8)**2+T(I+9)**2)*(T(I+8)+T(I+9))

```

100 CONTINUE

```

CD(120)=XK(120,12)*PI*R(1)**2/DELX
CD(121)=0.0
CD(122)=XK(122,14)*PI*(R(4)**2-R(3)**2)/DELX
CD(123)=0.0
CD(124)=XK(124,10)*PI*((R(5)+TMUF)**2-R(5)**2)/DELX
CD(125)=XK(125,4)*PI*((R(6)+THTR)**2-R(6)**2)/DELX
CD(126)=XK(126,10)*PI*((R(7)+TSTF)**2-R(7)**2)/DELX
CD(127)=XK(127,10)*PI*((R(8)+TSTF)**2-R(8)**2)/DELX
CD(128)=XK(128,10)*PI*((R(9)+TSTF)**2-R(9)**2)/DELX
CD(129)=XK(129,2)*PI*((R(10)+TWAL)**2-R(10)**2)/DELX
DO 200 I=120,210,10
CD(I+10)=CD(120)*XK(I+10,12)/XK(120,12)
CD(I+11)=0.0
CD(I+12)=CD(122)*XK(I+12,14)/XK(122,14)

```

```

CD(1+13)=0.0
CD(1+14)=CD(124)*XK(1+14,10)/XK(124,10)
CD(1+15)=CD(125)*XK(1+15,4)/XK(125,4)
CD(1+16)=CD(126)*XK(1+16,10)/XK(126,10)
CD(1+17)=CD(127)*XK(1+17,10)/XK(127,10)
CD(1+18)=CD(128)*XK(1+18,10)/XK(128,10)
CD(1+19)=CD(129)*XK(1+19,2)/XK(129,2)

```

200 CONTINUE

C
C
C

CONDUCTOR VALUES FOR REGION 2

```

CD(230) = XK(230,2)*PI*((R(10)+TVAL)**2-R(10)**2)/DELX
CD(231) = CD(230)*XK(231,2)/XK(230,2)
CD(232) = CD(230)*XK(232,2)/XK(230,2)
CD(233) = CD(230)*XK(233,2)/XK(230,2)
CD(234) = CD(230)*XK(234,2)/XK(230,2)
CD(235) = XK(235,2)*AM235/(R125-R126)
CD(236) = CD237*(T(126)**2+T(127)**2)*(T(126)+T(127))
CD(237) = CD237*(T(128)**2+T(127)**2)*(T(128)+T(127))
CD(238) = CD237*(T(129)**2+T(128)**2)*(T(129)+T(128))
CD(239) = CD237*(T(130)**2+T(129)**2)*(T(130)+T(129))
CD(240) = 0.0
CD(241) = CD(126)/2.*XK(241,10)/XK(126,10)
CD(242) = CD(126)/2.*XK(242,10)/XK(126,10)

CD(243) = CD(126)/2.*XK(243,10)/XK(126,10)
CD(244) = CD(126)/2.*XK(244,10)/XK(126,10)
CD(245) = XK(245,10)*PI*(R(8)+R(7))*TSTF/(R(8)-R(7))
CD(246) = XK(246,10)*PI*(R(9)+R(8))*TSTF/(R(9)-R(8))
CD(247) = XK(247,10)*PI*(R(10)+R(9))*TSTF/(R(10)-R(9))
CD(248) = CD(230)*XK(248,2)/XK(230,2)
CD(249) = CD237/2.*(T(118)**2+T(130)**2)*(T(118)+T(130))*F118/F128
CD(250) = CD237/2.*(T(119)**2+T(130)**2)*(T(119)+T(130))*F119/F128
CD(251) = XK(251,10)*PI*(R126+R(7))*TSTF/(R126-R(7))
CD(252) = XK(252,10)*PI*(R(10)+R126)*TSTF/(R(10)-R126)
CD(253) = CD(251)*XK(253,10)/XK(251,10)
CD(254) = CD(252)*XK(254,10)/XK(252,10)
CD(255) = CD(251)*XK(255,10)/XK(251,10)
CD(256) = CD(252)*XK(256,10)/XK(252,10)
CD(257) = CD(251)*XK(257,10)/XK(251,10)
CD(258) = CD(221)*XK(258,10)/XK(252,10)
CD(259) = XK(259,2)*AM259/(R126-R131)
CD(260) = XK(260,2)*PI*R131*.2*DELX/R131
CD(261) = CD261*((R(6)+THTR)**2-R(6)**2)*F116*
1 (T(116)**2+T(136)**2)*(T(116)+T(136))
CD(262) = CD261*((R(5)+TMUF)**2-R(5)**2)*F115*
1 (T(115)**2+T(136)**2)*(T(115)+T(136))
CD(263) = CD261*(R(4)**2-R(2)**2)*F113*
1 (T(113)**2+T(136)**2)*(T(113)+T(136))
CD(264) = CD261*R(1)**2*F111*
1 (T(111)**2+T(136)**2)*(T(111)+T(136))
IF(NTRANS.GT.0)CD(264)=0.

```

C
C
C

CONDUCTORS FOR REGION 3

$CD(265) = XK(265, 16) * PI * (R(4) + R(3)) / DELX$
 $CD(266) = CD(265) * XK(266, 16) / XK(265, 16)$
 $CD(267) = CD(265) * XK(267, 16) / XK(265, 16)$
 $CD(268) = CD(265) * XK(268, 16) / XK(265, 16)$
 $CD(269) = 0.0$
 $CD(270) = CD(124) * XK(270, 10) / XK(124, 10)$
 $CD(271) = CD(124) * XK(271, 10) / XK(124, 10)$
 $CD(272) = CD(124) * XK(272, 10) / XK(124, 10)$
 $CD(273) = CD(124) * XK(273, 10) / XK(124, 10)$
 $CD(274) = 0.0$
 $CD(275) = CD(126) * XK(275, 10) / XK(126, 10)$
 $CD(276) = CD(126) * XK(276, 10) / XK(126, 10)$
 $CD(277) = CD(126) * XK(277, 10) / XK(126, 10)$
 $CD(278) = CD(126) * XK(278, 10) / XK(126, 10)$
 $CD(279) = 0.0$
 $CD(280) = CD(127) * XK(280, 10) / XK(127, 10)$
 $CD(281) = CD(127) * XK(281, 10) / XK(127, 10)$
 $CD(282) = CD(127) * XK(282, 10) / XK(127, 10)$
 $CD(283) = CD(127) * XK(283, 10) / XK(127, 10)$
 $CD(284) = 0.0$
 $CD(285) = 1.89 * XK(285, 2) * PI * DELX$
 $CD(286) = CD(128) * XK(286, 10) / XK(128, 10)$
 $CD(287) = CD(128) * XK(287, 10) / XK(128, 10)$
 $CD(288) = 0.0$
 $CD(289) = CD(129) * XK(289, 2) / XK(129, 2)$
 $CD(290) = CD290 * (T(137)**2 + T(142)**2) * (T(137) + T(142))$
 $CD(291) = CD290 * (T(138)**2 + T(143)**2) * (T(138) + T(143))$
 $CD(292) = CD290 * (T(139)**2 + T(144)**2) * (T(139) + T(144))$
 $CD(293) = CD290 * (T(140)**2 + T(145)**2) * (T(140) + T(145))$
 $CD(294) = CD290 * (T(141)**2 + T(146)**2) * (T(141) + T(146))$
 $CD(295) = XK(295, 2) * PI * (R(7) + R(5)) * DELX * .50 / (R(7) + R(5))$
 $CD(296) = CD296 * (T(143)**2 + T(148)**2) * (T(143) + T(148))$
 $CD(297) = CD296 * (T(144)**2 + T(149)**2) * (T(144) + T(149))$
 $CD(298) = CD296 * (T(145)**2 + T(150)**2) * (T(145) + T(150))$
 $CD(299) = CD296 * (T(146)**2 + T(151)**2) * (T(146) + T(151))$
 $CD(300) = CD(295) * 2.0 * XK(300, 2) / XK(295, 2)$
 $CD(301) = CD301 * (T(148)**2 + T(153)**2) * (T(148) + T(153))$
 $CD(302) = CD301 * (T(149)**2 + T(154)**2) * (T(149) + T(154))$
 $CD(303) = CD301 * (T(150)**2 + T(155)**2) * (T(150) + T(155))$
 $CD(304) = CD301 * (T(151)**2 + T(156)**2) * (T(151) + T(156))$
 $CD(305) = CD(285) * XK(305, 2) / XK(285, 2)$
 $CD(306) = CD306 * (T(153)**2 + T(157)**2) * (T(153) + T(157))$
 $CD(307) = CD306 * (T(154)**2 + T(158)**2) * (T(154) + T(158))$
 $CD(308) = CD306 * (T(155)**2 + T(159)**2) * (T(155) + T(159))$
 $CD(309) = CD306 * (T(156)**2 + T(160)**2) * (T(156) + T(160))$
 $CD(310) = XK(310, 2) * PI * (R(10) + R(9)) * DELX / (R(10) + R(9))$
 $CD(311) = CD311 * (T(159)**2 + T(162)**2) * (T(159) + T(162))$
 $CD(312) = CD311 * (T(160)**2 + T(163)**2) * (T(160) + T(163))$

```

CD(313)=CD(124)*XK(313,10)/XK(124,10)
CD(314)=CD(126)*XK(314,10)/XK(126,10)
CD(315)=CD(127)*XK(315,10)/XK(127,10)
CD(316)=CD(128)*XK(316,10)/XK(128,10)
CD(317)=CD(129)*XK(317,2)/XK(129,2)
CD(261)=0.0
CD(262)=0.0
CD(263)=0.0
CD(264)=0.0
IF(NTRANS.EQ.0) GO TO 250
DO 245 I=1,NTRANS
K=1+10*(1+I-NTRANS)
J=1+10*(I-1)
CD(K)=CD21*(T(J)**2+T(137)**2)*(T(J)+T(137))
245 CONTINUE
250 CONTINUE
C THE FOLLOWING CONDUCTORS ARE FOR THE HELIUM GAS CONTAINED IN
C THE MUFFLE
A1=2.*PI*R(1)*DELX
AVG1=(A1+A2)/2.
XDIS1=R(2)-R(1)
AVG2=(A4+A5)/2.
XDIS2=R(5)-R(4)
CONHE1=XKHE*AVG1/XDIS1
CONHE2=XKHE*AVG2/XDIS2
DO 2001 I=318,329
CD(I)=CONHE1
2001 CONTINUE
DO 2002 I=330,346
CD(I)=CONHE2
2002 CONTINUE
C CONDUCTION BETWEEN HOT AND COLD HEAT PIPES
NLAYER=30
ELAYER=EW
CD(347)=PI*(R(4)**2-R(2)**2)*SIGMA*(ELAYER/(2.-ELAYER))/NLAYER*
+ (T(3)**2+T(141)**2)*(T(3)+T(141))
RETURN
END

```

```
SUBROUTINE TSP(C,CD,NCTRL,M,NNDS,NPR,PRIN,TIM,IPT,TSCON,DT)
DIMENSION C(200),CD(500),M(200,15)
SMAL = 1.0E30
DO 100 I = 1,NNDS
CAP = C(I)
IF(CAP) 100,100,400
400 CONTINUE
NCOND = M(I,1)
CSUM = 0.0
DO 200 J = 1,NCOND
IC = J*2+1
NC = M(I,IC)
CSUM = CSUM + CD(NC)
200 CONTINUE
STEST = CAP/CSUM
IF(STEST.GE.SMAL) GO TO 300
SMAL = STEST
NCTRL = I
300 CONTINUE
100 CONTINUE
DT = SMAL/TSCON
TPR = FLOAT(NPR)*PRIN
TT1 = TIM+DT
IPT = 0
IF(TPR.GT.TT1) GO TO 500
DT = TPR-TIM
IPT = 1
500 CONTINUE
RETURN
END
```

```
SUBROUTINE INTP(X,N,Y)
COMMON/DTA/CC(40,51)
J = CC(N,1)
J1 = J*2
XLST = CC(N,2)
XHST = CC(N,J1)
IF(X.EQ.XLST) GO TO 700
IF(X.LT.XLST) GO TO 100
IF(X.GT.XHST) GO TO 200
JT = 4
500 CONTINUE
IF(CC(N,JT)-X) 300,600,400
300 CONTINUE
JT = JT+2
GO TO 500
400 CONTINUE
N1 = JT-2
N2 = N1+1
N3 = JT
N4 = N3+1
GO TO 900
600 CONTINUE
NT = JT+1
Y = CC(N,NT)
GO TO 1000
700 CONTINUE
Y = CC(N,3)
GO TO 1000
100 CONTINUE
N1 = 2
N2 = 3
N3 = 4
N4 = 5
GO TO 900
200 CONTINUE
N1 = 2*J-2
N2 = N1+1
N3 = 2*J
N4 = N3+1
900 CONTINUE
SL = (CC(N,N4)-CC(N,N2))/(CC(N,N3)-CC(N,N1))
Y = CC(N,N2)+SL*(X-CC(N,N1))
1000 CONTINUE
RETURN
END
```

00

```
      SUBROUTINE DATA
      COMMON/DTA/CC(40,51)
701  FORMAT(2I5)
702  FORMAT(2E10.0)
703  FORMAT(1H1,10X,11HT A B L E S)
704  FORMAT(1H0,5X,2I5)
705  FORMAT(1H0,5X,2E12.5)
      WRITE(6,703)
100  CONTINUE
      READ(5,701)N,J
      WRITE(6,704)N,J
      IF(N.EQ.0) GO TO 300
      CC(N,1) = J
      DO 200 I = 1,J
      READ(5,702) A,B
      WRITE(6,705)A,B
      K1 = I*2
      K2 = K1+1
      CC(N,K1) = A
      CC(N,K2) = B
200  CONTINUE
      GO TO 100
300  CONTINUE
      RETURN
      END
```

Appendix B
LISTING OF INPUT DATA CARDS FOR SAMPLE PROBLEM

	100.	50	100	2.0	100.	540.	540.
2000.	2000.		2000.	2000.			
5							
1	9						
0.0		.055					CP - STAINLESS STEEL
250.		.085					
500.		.108					
750.		.123					
1000.		.133					
1500.		.142					
2000.		.152					
2500.		.164					
3000.		.177					
2	9						
0.0		4.8					K - STAINLESS STEEL
250.		6.7					
500.		8.3					
750.		9.8					
1000.		11.1					
1500.		13.2					
2000.		15.2					
2500.		17.1					
3000.		19.0					
3	10						
0.0		.016					CP - ALUMINA
200.		.022					
400.		.025					
600.		.027					
800.		.028					
1000.		.029					
1500.		.030					
2000.		.031					
2500.		.032					
3000.		.032					
13	2						
100.		.100					CP-HEAT PIPE
4000.		.100					
14	2						
100.		1.0					K-HEAT PIPE
4000.		1.0					
15	2						
100.		.100					CP-COLD HEAT PIPE
4000.		.100					
16	2						
100.		1.0					K-COLD HEAT PIPE
4000.		1.0					

B-1

4	10		
0.0	22.5	K - ALUMINA	
200.	17.8		
400.	14.8		
600.	9.3		
800.	7.2		
1000.	5.7		
1500.	3.9		
2000.	3.2		
2500.	3.2		
3000.	3.1		
5	2		
100.	.036	CP - TUNGSTEN	
3500.	.036		
6	10		
100.	220.0	K - TUNGSTEN	
200.	130.0		
300.	105.		
500.	93.		
1000.	79.		
1500.	73.		
2000.	68.		
2500.	65.		
3000.	62.		
3500.	59.5		
7	12		
0.	.046	CP - MOLY	
250.	.057		
500.	.062		
750.	.064		
1000.	.065		
1250.	.066		
1500.	.067		
2000.	.070		
2500.	.076		
3000.	.083		
3500.	.091		
4000.	.100		
8	12		
0.0	84.	K - MOLY	
250.	82.5		
500.	79.		
750.	76.5		
1000.	74.5		
1250.	72.5		
1500.	69.5		
2000.	63.5		
2500.	57.		
3000.	51.0		
3500.	45.5		
4000.	42.		

9	2		
0.0		.100	CP - TA-10W
4000.		.100	
10	5		
0.0		33.5	K - TA-10W
2400.		33.5	
3200.		29.5	
4000.		26.	
4800.		22.5	
11	2		
100.		.036	CP - TUNGSTEN(SAMPLE)
3500.		.036	
12	10		
100.		220.0	K - TUNGSTEN
200.		130.0	
300.		105.	
500.		93.	
1000.		79.	
1500.		73.	
2000.		68.	
2500.		65.	
3000.		62.	
3500.		59.5	