# Preflight Transient Dynamic Analyses of B-52 Aircraft Carrying Space Shuttle Solid Rocket Booster Drop-Test Vehicle 

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AUTH: A/NO, W. L;' B/GCHUSER, L. S.
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    Woffett Field, Gali, fvall.NTLS GAP: HC AOZ/mF A01
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# Preflight Transient Dynamic Analyses of B-52 Aircraft Carrying Space Shuttle Solid Rocket Booster Drop-Test Vehicle 

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PREFLIGHT TRANSIENT DYNAMIC ANALYSES OF B-52 AIRCRAFT CARRYING SPACE SHUTTLE SOLID ROCKET BOOSTER DROP-TEST VEHICLE

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| $\theta$ | $=\mathrm{B}-52$ nose-up ang1e, deg |
| :---: | :---: |
| ${ }^{0} x, 0^{\prime}, 0_{z}$ | $\begin{aligned} = & \text { rotation about the } \mathrm{x}-, \mathrm{y}-\text {, or } \\ & \mathrm{z} \text {-axis, rad } \end{aligned}$ |
| $\mu$ | ```= coefficient of friction between landing gear tires and the ground surface``` |
| $\tau$ | $=$ delay time, sec |
| T0 | ```= braking force ramp rise time, sec``` |
| ${ }^{\omega} \mathrm{P}$ | $=$ vibration frequencies of pylon in pitching motion, $1 / \mathrm{sec}$ |
| $\omega_{S}$ | $=$ vibration frequencies of pylon in side motion, $1 / \mathrm{sec}$ |
| $\omega_{\mathrm{Y}}$ | $=$ vibration frequencies of pylon in yaw motion, $1 / \mathrm{sec}$ |
| ['], ["] | $\begin{aligned} = & \text { time derivatives, } 1 / \mathrm{sec}, \\ & 1 / \mathrm{sec}^{2} \end{aligned}$ |

## Introduction

The performance of the decelerator system (1.e., the main parachute) of the Space Shuttle solid-rocket booster (SRB) was investigated by using a scale model drop-test vehicle. (DTV). The DTV, which carried only one main parachute (the actual SRB has three main parachutes), was attached to a pylon on the NASA B-52 carrier aircraft by means of one front hook and two rear hooks (see Figs. 1 and 2). The DTV was then carried to high altitude and released from the $B-52$ to test the deployed $S R B$ main parachute.

In the past captive flights (i.e., no releasing of the DTV from the $B-52$ ), the $B-52$ pylon front hook was loaded up to considerable levels. The most serious instances were the quarter-elevator throws (the front hook load reached $89 \%$ of the limit load of $37,700 \mathrm{lb}$ ), ground turns (the front hook load reached $79 \%$ of the limit load), and landings (the front-hook load reached $76 \%$ of the limit load). Because of these recorded high loading levels of the front hook, there is concern about possible overloadings of the front hook in an upcoming new series of DTV tests.

In order to reduce the pitching moment of inertia and thereby reduce the front-hook dynamic loadings, the DTV was shortened slightly. Before testing the revised DTV system, it was necessary to perform preflight transient dynamic analyses of the entire $\mathrm{B}-52 / \mathrm{DTV}$ system, and to estimate the range of the front-hook loadings under conditions imposed by various maneuvers of the B-52. This paper concerns the transient dynamic analysis of the B-52/DTV system when either landing or braking on aborted takeoff runs. The front- and the rearhook loading levels calculated under different landing and braking conditions can then be used to establish the safe maneuver envelopes for the $B-52$ carrier aircraft.

## Transient Dynamic Analyses

## NASTRAN Finite-Element Modeling

The NASA Structural Analysis (NASTRAN) finiteelement computer program ${ }^{1}$ was used in the transient dynamic analysis of the B-52/DTV system. As shown in Fig. 2, the B-52 carrier structure was modeled by using CBAR elements (uniform cross section bar element) for carrying the structural stiffnesses; the B-52 structural masses were lumped at most of the grid points, and were modeled by using C $\emptyset$ NM1 elements (concentrated mass elements of type 1) for carrying the structural inertia effect. Landing gears were not modeled. Grid points 9, 17, 26, 43, 62, 64,71 , and 73 are massless points; therefore, they were removed in the dynamic analyses with $\emptyset$ MIT cards. Figure 3 shows the NASTRAN model of the B-52 carrying the DTV. The front and rear landing gears are located at grid points 6 and 15 , respectively. The center of gravity of the B-52 varies between grid points 12 and 13. Because of very high stiffnesses, elements lying between grid-points 7 and 9,9 and 10,16 and 17, 23 and 26 , and 24 and 43, were modeled as rigid elements, using CRIGD1 constraints (rigid constraints of type 1).

The vertical tail has extremely high vertical bending stiffness (about the $y$-axis); therefore, CRIGD2 constraints (rigid constraints of type 2) were used to force the entire vertical tail to rotate as a whole about the $y$-axis. Namely, rotations about the $y$-axis of grid-points 27 through 35 were made identical to those of grid-point 26 . Also, because of very high lateral bending stiffnes (about $z$ ) of the wing roots, rotations about the $z$-axis of grid-points 67 and 68 were set equal to that of grid-point 9 through the use of CRIGD2 constraints. The reason for using the above rigid constraints CRIGD1 and CRIGD2 was to avoid the solution errors caused by using very large values of stiffnesses for the CBAR elements.

For each C $\varnothing \mathrm{NM}$ element, a local rectangular Cartesian coordinate system ( $\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}$ ) (with origin located at the associated mass grid point) was defined by using C $\emptyset$ RD2R cards. At each mass grid point, the mass matrix [M] was set up in the ( $\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}$ ) system for input to the C $\varnothing \mathrm{NM} 1 \mathrm{card}$. Because the centers of mass of most of the C $\varnothing$ NMI elements were offset from the associated grid points, some of the off-diagonal terms in [M] shown below were nonzero:

$$
[M]=\frac{1}{g}\left[\begin{array}{cccccc}
W & 0 & 0 & 0 & x_{3} W & -x_{2} W  \tag{1}\\
& W & 0 & -x_{3} W & 0 & x_{1} W \\
& & W & x_{2} W & -x_{1} W & 0 \\
& & I_{11} & -I_{12} & -I_{13} \\
{[\text { symmetry }]} & & I_{22} & -I_{23} \\
& & & & I_{33}
\end{array}\right]
$$

where W is the weight of the $\mathrm{C} \varnothing \mathrm{NM}$ element (or CøNM2 element described below), and $I_{1 j}$
( $1, j=1,2,3$ ) are the weight moments of inertia of the C $\varnothing \mathrm{NM} 1$ element (or C $\varnothing \mathrm{NM} 2$ element) referred to the ( $\mathrm{x}_{1}, \mathrm{x}_{2}, \mathrm{x}_{3}$ ) system. The DTV was modeled by using one C $\varnothing \mathrm{NM} 2$ element (concentrated mass element of type 2) with the DTV center of mass
located at grid point 85 (no mass offset). The mass matrix [Eq. (1)] of this C $\varnothing \mathrm{NM} 2$ element is based on a local rectangular Cartesian coordinate system, with the origin at grid-point 85, defined by using a $C \not \subset \mathrm{RD} 2 \mathrm{R}$ card. The mass of the pylon was lumped at grid-point 70 and was represented by a C $\emptyset \mathrm{NM} 1$ element whose mass matrix [Eq. (1)] is based on a local rectangular Cartesian coordinate system, with origin at grid-point 70, defined by a $C \emptyset R D 2 R$ card. In the data input to the mass matrix [Eq. (1)] for all the mass elements C $\varnothing \mathrm{NM} 1$ and C $\varnothing$ NM2, weight was used, which was converted into mass through the use of a PARAM/WTMASS card.

In modeling the pylon, two cases were considered: a rigid pylon and an elastic pylon. For the case of the rigid pylon, grid-point 85 was rigidly attached to grid-point 70 through a CRIGD1 rigid constraint. For the elastic pylon, a CRIGD2 card was used to rigidly couple the motions in the $x$ - and $z$-directions of grid-point 85 to those of grid-point 70 (i.e., perfectly rigid in $z$-extension, infinite shear stiffness in the $x-z$ plane). The rest of the degrees of freedom were set free (see Fig. 4). For the yaw motion, a torsional spring CELAS2 element was attached between grid-points 70 and 85 (Fig. 4a) to allow spring resistance only in yaw motion (sixth degree of freedom). For the roll motion (or side motion), a CRIGDI rigid element was attached between gridpoints 70 and 87 (grid-point 87 is coincidental with grid-point 85). A spring CELAS2 e1ement was then attached between grid-points 85 and 87 , permitting the motion of grid-point 85 only in $y$-direction (second degree of freedom) (Fig. 4b). Lastly, the pitching elastic resistance was modeled by connecting a bending spring element CELAS2 between grid-points 70 and 85 , permitting only the pitching motion (fifth degree of freedom) of the pylon (Fig. 4c). The spring constants for the above three CELAS2 elements were calculated from the measured vibration frequencies (derived from $\mathrm{X}-15$ reports) using the following equations:

$$
\begin{align*}
& \mathrm{K}_{\mathrm{Y}}=\frac{\mathrm{I}_{\mathrm{Y}} \omega_{\mathrm{Y}}^{2}}{\mathrm{~g}} ; \quad \mathrm{I}_{\mathrm{Y}} \cong \mathrm{I}_{\mathrm{P}}  \tag{2}\\
& \mathrm{~K}_{\mathrm{S}}=\mathrm{mw}_{\mathrm{S}}^{2}  \tag{3}\\
& \mathrm{~K}_{\mathrm{P}}=\frac{\mathrm{I}_{\mathrm{P}} \omega_{\mathrm{P}}^{2}}{\mathrm{~g}} \tag{4}
\end{align*}
$$

where $K_{Y}, K_{S}$, and $K_{p}$ are, respectively, spring constants for yaw, side, and pitching motions; m 1s the mass of the DTV; $I_{P}$ and $I_{Y}$ are, respectively, the pitching and the yawing moments of inertia of DTV; and $\omega_{Y}, \omega_{S}$, and $\omega_{p}$ are, respectively, angular vibration frequencies in yaw, side, and pitching motions.

The entire $\mathrm{B}-52 / \mathrm{DTV}$ NASTRAN model has 88 grid points, 78 CBAR elements, 7 CRIGD1 elements, 76 C $\varnothing \mathrm{NM} 1$ elements, and $1 \mathrm{C} \varnothing \mathrm{NM} 2$ element. The NASTRAN model free system is supported at gridpoints 12 (with components, $1,2,3$, and 4 constrained) and 13 (with components 2 and 3 constrained) through the use of a SUPORT card which is to remove the stress-free, rigid-body motion component from the free system.

Forcing Functions
In the landing analyses, four cases of landing conditions were considered, and are shown below:

| Case | Sink speed <br> $V_{S}, f t / s e c$ | Nose-up <br> angle, deg |
| :---: | :---: | :---: |
| Front landing gear <br> touchdown delay time <br> $\tau$, sec |  |  |
| 2 | 3 | 3 |

The landing-gear reaction forces (obtained from the manufacturer) at the fuselage for the case $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$, are shown in Figs. 5 and 6. The landing-gear reaction forces for the remaining three cases are similar and, therefore, are not shown. Those landing forces are to be applied at grid-points 6 (front-landing-gear attach point) and 15 (rear-landing-gear attach point) of the NASTRAN model. Those forcing functions (e.g., Figs. 5 and 6) were converted into tables by using TABLED 1 cards for input to the NASTRAN model. The TABLEDI cards were used in conjunction with the following cards in the Bulk Data Deck:

TLøAD1 = Define a time-dependent dynamic load of the form $\mathrm{AF}(\mathrm{t}-\tau)$

DAREA $=$ Define the dynamic load scale factor $A$
DELAY $=$ Define the dynamic load delay time $\tau$
DLøAD $=$ Combination of dynamic load
Also, in the Case Control Deck, the dynamic load set must be defined through $\operatorname{DLD} A D=n$, where n is the set identification number. The initial. velocity (sink velocity, in the negative z-direction) of the NASTRAN model was defined by using TIC cards in the Bulk Data Deck for all the grid points excluding the massless grid points and the rigid element grid points. In the Case Control Deck, the transient initial condition set must be selected through the use of $I C=n^{\prime}$ where $n^{\prime}$ is the set identification number. The integration time-steps and the solution output time-intervals were specified by using a TSTEP card in the Buik Data Deck. The time-delay effect ( $\tau$ ) of the front-landing-gear forcing function can be handled in two ways:

1) Without using the Delay card, construct the front-landing-gear forcing function table TABLEDI in such a way that the delay time is absorbed into the table
2) Using the DELAY card, construct the front-landing-gear forcing function table tableD1 by shifting the time origin to the force rising time [i.e., by using ( $t-\tau$ ) as the time scale]

In the braking analyses, the transient dynamic forcing functions (landing-gear braking force or braking moment) for input to the NASTRAN model are shown in Fig. 7. The coefficient of friction $\mu$ between the landing-gear tires and the ground
surface was taken to be 0.55 . Brakes were applied when the $B-52$ was taxiling at 135 knots (2734.25 in./sec). The lengths of braking force famp-rise-time to (Fig. 7) considered in the $^{\text {( }}$ braking analyses are

| Case | $0, \mathrm{sec}$ |
| :---: | :---: |
| 1 | 0.2 |
| 2 | 0.5 |
| 3 | 1.0 |

The braking forces and the associated braking moments at the landing-gear attach points 6 and 15 are given by

$$
\begin{align*}
F_{6} & =\mu W_{6}  \tag{5}\\
F_{15} & =\mu W_{15}  \tag{6}\\
M_{6} & =h_{6} F_{6}  \tag{7}\\
M_{15} & =h_{15} F_{15} \tag{8}
\end{align*}
$$

where $W_{i}(i=6,15)$ is the weight on the landing gear at grid-point $i$, where $F_{i}(i=6,15)$ and $M_{i}(i=6,15)$ are, respectively, the braking force and the braking momenc at grid-point $i$, and where $h_{1}(i=6,15)$ is the length of the landing gear. The braking forcing functions and the initial conditions were input into the NASTRAN model in a way similar to that used in the landing analyses. DELAY cards may be omitted since there is no time delay of the front-1anding-gear forcing function.

## Input Data for Structural Mode1

The grid-point coordinates, nodal weight, and stiffness data for input to the B-52 NASTRAN model and the inertia data for input to Eq . (1) were taken from Ref. 2. Other input data are
$E=10 \times 10^{6} 1 \mathrm{~b} / \mathrm{in}^{2}$
$G=4 \times 10^{6} \mathrm{lb} / \mathrm{in} .{ }^{2}$
B-52 structural damping coefficient
$=0.015 \mathrm{lb} \cdot \mathrm{sec} / \mathrm{in}$.

DTV weight $W_{\text {DTV }}=m g=49,000 \mathrm{lb}$
DTV pitching (or yawing) moment of inertia $I_{P}=1.4591 \times 10^{9} 1 \mathrm{~b} \cdot \mathrm{in} .^{2}$

DTV rolling moment of inertia
$\mathrm{I}_{\mathrm{R}}=4.5537 \times 10^{7} \mathrm{ib} \cdot \mathrm{in}^{2}$
Pylon weight $=1170 \mathrm{1b}$

Pylon mass matrix:

The peak value of $V_{C L}$ reached only $38 \%$ of the limit value of $57,600 \mathrm{lb}$. The time-histories of the hook loads for the rest of the landing conditions $-\mathrm{V}_{\mathrm{S}}=3 \mathrm{ft} / \mathrm{sec}, \theta=6^{\circ}$; $\mathrm{V}_{\mathrm{S}}=3 \mathrm{ft} / \mathrm{sec}$, $\theta=0^{\circ}$ (simultaneous touchdown of the front and rear landing gears); and $V_{S}=6 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$ are quite similar to the case in which $V_{S}=3 \mathrm{ft} / \mathrm{sec}, 0=3^{\circ}$; therefore, they are not shown here. The calculated peak values of $\mathrm{V}_{\mathrm{A}}$, $\mathrm{V}_{\mathrm{CL}}, \mathrm{S}_{\mathrm{A}}, \mathrm{S}_{\mathrm{CL}}$, and $\mathrm{D}_{\mathrm{CL}}$ for the elastic pylon for the four landing conditions are listed in Table 1; the peak values of $V_{A}$ and $V_{C L}$ are plotted in Figs. $17\left(V_{\mathrm{A}}\right)$ and $18\left(\mathrm{~V}_{\mathrm{CL}}\right)$. The landing condition $\left(V_{S}=6 \mathrm{ft} / \mathrm{sec}, 0=3^{\circ}\right)$ induced the highest peak values of $V_{A}\left(86 \%\right.$ of 1 imit 1 oad) and $V_{C L}$ ( $48 \%$ of limit load)... Figures 19 and 20, respectively, show $\ddot{z}$ and $\ddot{\theta}_{\mathrm{y}}$ at grid -point 85 (DTV) for the braking condition at $\tau_{0}=0.2 \mathrm{sec}, \mu=0.55$, for both elastic and rigid pylons. The $\ddot{z}$ for the braking is insignificant when compared with $\ddot{\theta}_{y}$ which has about 3 cycles/sec of oscillation for both elastic and rigid cases.

All the calculated time-histories for the hook loads (see Appendix) are shown in Figs. 21-25 for the braking condition at $\tau_{0}=0.2 \mathrm{sec}$, $\mu=0.55$, for both elastic and rigid pylons. Notice that the rear-hook drag load DCL (Fig. 25) reflects the shape of braking forcing function. The time histories of the hook loads for the other two braking conditions $-\tau_{0}=0.5 \mathrm{sec}, \mu=0.55$ and $\mathrm{t}_{0}=1.0 \mathrm{sec}, \mu=0.55$ - are similar to the case $\tau_{0}=0.2 \mathrm{sec}$; therefore, they are not shown. The braking tends to decrease the mean value of $V_{A}$ from its static value and increase the mean value of $V_{C L}$ from its static value. However, the peak values of $V_{C L}$ stay well below the limit value. The peak values of $V_{A}, V_{C L}, S_{A}, S_{C L}$, and $\mathrm{D}_{\mathrm{CL}}$ for the elastic pylon for the three braking conditions are tabulated in Table 1, and the peak values of $\mathrm{V}_{\mathrm{A}}, \mathrm{V}_{\mathrm{CL}}$, and $\mathrm{D}_{\mathrm{CL}}$ are plotted in Figs. 26-28. The braking condition $\tau_{0}=0.2 \mathrm{sec}$, $\mu=0.55$ gives the highest peak values of $V_{A}$ ( $39 \%$ of the 1 imit load), $V_{C L}$ ( $46 \%$ of the 1 imit $1 \mathrm{oad}), \mathrm{S}_{\mathrm{A}}$ ( $89 \%$ of the 1 imit 1 oad ), $\mathrm{S}_{\mathrm{CL}}$ ( $41 \%$ of the limit load), and $\mathrm{D}_{\mathrm{CL}}$ ( $53 \%$ of the limit load). The peak values of $\mathrm{V}_{\mathrm{A}}$ for all three cases stay close to its static value.

## Summary

The NASTRAN finite-element computer program was used in transient dynamic analyses of a $\mathrm{B}-52$ aircraft carrying the Space Shuttle solid-rocket booster drop-test vehicle when either landing or braking. All the hook loads of the B-52 pylon were calculated for different landing and braking conditions. Both elastic and rigid pylons were considered. For landing, it was found that the vertical hook loads were more sensitive to the landing conditions than side and drag loads. The landing condition ( $6-\mathrm{ft} / \mathrm{sec}$ sink rate, $3^{\circ}$ nose-up) was found to be the worst case; it caused the peak value of $\mathrm{V}_{\mathrm{A}}$ to reach $86 \%$ of its limit load, and the peak value of $V_{\text {CL }}$ to reach $48 \%$ of its limit load. For braking, the peak values of $V_{A}$ and $V_{C L}$ are relatively insensitive to the braking conditions, but the side loads of both front and rear hooks oscillate considerably about their static values (i.e., zero). The rear-hook drag load was quite sensitive to the braking conditions. Braking tends to decrease the front-hook mean vertical load, and slightly increase the rear-hook
vertical load from their respective static values.

The information on the hook loads can now be used as a basis to establish safe maneuver envelopes for the B-52 carrying the SRB/DTV when landing or braking.

## Appendix: Equations for Hook Loads

The following equations for hook loads were developed in accordance with the dimenaions shown in Figs. 29 through 31.

1) Front-hook dynamic vertical load $V_{A D}$ (considering moment about point A) (Fig, 30a):

$$
\begin{align*}
\mathrm{V}_{\mathrm{AD}}= & \frac{1}{211.0}(\mathrm{~m} \ddot{z}(60.8)-\mathrm{m} \ddot{x}(27.35-17.5) \\
& \left.+\left\{I_{\mathrm{p}}+\mathrm{m}\left[(60.8)^{2}+(27.35-17.5)^{2}\right]\right\} \ddot{\theta}_{\mathrm{y}}\right) \tag{A1}
\end{align*}
$$

2) Front-hook static vertical load $V_{A S}$ :

$$
\begin{equation*}
\mathrm{v}_{\mathrm{AS}}=\frac{60.8}{211.0} \mathrm{~W}_{\mathrm{DTV}} ; \quad \mathrm{W}_{\mathrm{DTV}}=\mathrm{mg} \tag{A2}
\end{equation*}
$$

3) Front-hook total vertical load $V_{A}$ :

$$
\begin{equation*}
\mathrm{V}_{\mathrm{A}}=\mathrm{V}_{\mathrm{AD}}+\mathrm{v}_{\mathrm{AS}} \tag{A3}
\end{equation*}
$$

4) Front-hook side load $S_{A}$ (considering equivalent moment about point 0 ) (Fig. 30b):

$$
\begin{equation*}
S_{A}=\frac{1}{220.29}\left\{m \ddot{y}(60.8)-\left[I_{P}+m(60.8)^{2}\right] \ddot{\theta}_{z}\right\} \tag{A4}
\end{equation*}
$$

where 220.29 is the equivalent moment arm (from Ref. 3).
5) Rear-hook dynamic vertical load $V_{\text {CLD }}$ (considering moment about point CR) (Fig. 31):

$$
\begin{align*}
v_{C L D}= & \frac{1}{62.624}\left(\left(m \ddot{z}-v_{A D}\right)(31.312)-m \ddot{y}(9.85)\right. \\
& -S_{A}(16.3125) \\
& \left.-\left\{I_{R}+m\left[(9.85)^{2}+(31.312)^{2}\right]\right\} \ddot{\theta}_{x}\right) \tag{A5}
\end{align*}
$$

6) Rear-hook static vertical load $V_{C L S}$ :

$$
\begin{equation*}
\mathrm{V}_{\mathrm{CLS}}=\frac{1}{2} \frac{150.2}{211.0} \mathrm{~W}_{\mathrm{DTV}} \tag{A6}
\end{equation*}
$$

7) Rear-hook total vertical load $V_{C L}$ :

$$
\begin{equation*}
v_{C L}=v_{C L D}+v_{C L S} \tag{A7}
\end{equation*}
$$

8) Rear-hook side load $S_{C L}$ :

$$
\begin{equation*}
S_{C L}=m \ddot{y}-S_{A} \tag{A8}
\end{equation*}
$$

9) Rear-hook drag load $\mathrm{D}_{\mathrm{CL}}$ (considering equivalent moment about point 0) (Fig. 30b):
$D_{C L}=\frac{1}{1484.475}\left\{\left[\mathrm{I}_{\mathrm{P}}+\mathrm{m}(60.8)^{2}\right] \ddot{\theta}_{z}-m \ddot{y}(60.8)\right\}$

$$
\begin{equation*}
+\frac{1}{2} m \ddot{x} \tag{A9}
\end{equation*}
$$

where 1484.475 is the equivalent moment arm (from Ref. 3).

## References

${ }^{1}$ "NASTRAN Theoretical Manual, NASTRAN Users' Manual," Computer Software Management and Information Center (COSMIC), Ballows Halls, University of Georgia, Atlanta, Ga., 1978.
${ }^{2}$ Hu11, D. L. and Roger, K. L., "B-52E CCV Flight Test Data Applicable to Parameter Estimation," Air Force Flight Dynamics Laboratory TR-75-131, 1975.
${ }^{3}$ Quade, D. A., "Load and Dynamic Assessment of B-008 Carrier Alrcraft for Configuration 1 and 2 Space Shuttle Solid Rocket Booster Deceleration Subsystem Drop Test Vehicles. Vol. IV, Pylon Load Data Method 2," Contract No. NA58-31805, Boeing Aircraft Company Report D3-1.220-1, 1977.

Table 1 Peak hook loads associated with different landing and braking conditions



Fig. 1 Geometry of Space Shuttle solid-rocket booster drop-test vehicle (DTV) attached to B-52 pylon; view looking inboard at right side of $B-52$ and SRB/DTV.


Fig. 2 B-52 structure represented by bar elements and concentrated mass elements.


Fig. 3 B-52 NASTRAN model.


Fig. 4 Modeling of elastic pylon. a) Yaw, b) roll, c) pitch.


Fig. 5 Combined front landing gear force: ${ }^{v_{S}}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 6 Combined rear landing gear force: $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 7 Landing-gear braking force (or moment) curve.


Fig. 8 Transient dynamic response of $\mathrm{B}-52$ to upward triangular pulse ( 1.5 g ) at front and rear landing gears: $t=1 \mathrm{sec}$, single-point constraint at grid points 12 and 13 .


Fig. 9 Transient dynamic response of $\mathrm{B}-52$ to upward triangular pulse ( 1.5 g ) at front and rear landing gears: $t=2.1 \mathrm{sec}$, single-point constraint at grid points 12 and 13 .


Fig. 10 z-acceleration at grid point 85 (DTV):
landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 11 Angular acceleration about $y$-axis at grid point 85 (DTV): landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 12 Front-hook vertical load versus time: landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 13 Front-hook side load versus time: landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 14 Rear-hook vertical load versus time: landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, \theta=3^{\circ}$.


Fig. 15 Rear-hook side load versus time: landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, 0=3^{\circ}$.


Fig. 16 Rear-hook drag load versus time: landing, $V_{S}=3 \mathrm{ft} / \mathrm{sec}, 0=3^{\circ}$.


Fig. 17 Variation of front-hook peak vertical load with landing conditions.


Fig. 18 Variation of rear hook peak vertical load with landing conditions.


Fig. 19 z-acceleration at grid point 85 (DTV): braking, $\tau_{o}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 20 Angular acceleration about $y$-axis at grid point 85 (DTV): braking, $\tau_{o}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 21 Front-hook vertical load versus time: braking, $\tau=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 22 Front-hook side load versus time: braking, $\mathrm{T}_{0}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 23 Rear-hook vertical load versus time: braking, $\tau_{o}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 24 Rear-hook side load versus time: braking, $\mathrm{T}_{\mathrm{o}}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 25 Rear-hook drag load versus time: braking, $\tau_{0}=0.2 \mathrm{sec}, \mu=0.55$.


Fig. 26 Variation of front-hook peak vertical load with braking condition.


Fig. 27 Variation of rear-hook peak vertical load with braking condition.


Fig. 28 Variation of rear-hook peak drag load with braking condition.


Fig. 29 Locations of DTV and front and rear hooks: Eq. (A1).


V: VERTICAL LOAD
S: SIDE LOAD
D: DRAG LOAD

Fig. 30 Locations of DTV and front and rear hooks: Eqs. (A4) and (A9). a) Side view; b) Top view.


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[^0]:    *For sale by the National Technical Information Service, Springfield, Virginia 22161.

