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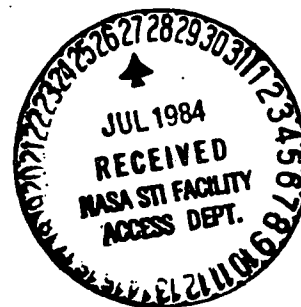
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ABSTRACT

A relatively simple method has been developed to optimize the location, temperature, and heat dissipation rate of each cooled shield inside an insulation layer. The method is based on the minimization of the entropy production rate which is proportional to the heat leak across the insulation. The results show that the maximum number of shields to be used in most practical applications is three. However, cooled shields are useful only at low values of the overall, cold wall to hot wall absolute temperature ratio. The performance of the insulation system is relatively insensitive to deviations from the optimum values of temperature and location of the cooling shields.

Design curves are presented for rapid estimates of the locations and temperatures of cooling shields in various types of insulations, and an equation is given for calculating the cooling loads for the shields.

NOMENCLATURE

A	Area of heat flow, m^2
C_p	Specific heat of the boiloff vapor, $kJ/kg \cdot K$
D	Functional defined by Eq. (14)
F	Functional defined by Eq. (13)
h_{fg}	Latent heat of vaporization of the boiloff liquid, kJ/kg
k	Thermal conductivity, $W/m \cdot K$; with subscripts, coefficients in Eq. (1)
L	Overall thickness of insulation, m^*
m, n	Exponents in conductivity function, Eq. (1)
P	T_S/T_C , temperature ratio
q	Heat flow rate, W
R	T_C/T_H , overall temperature ratio
s	Dimensionless entropy production rate defined by Eq. (5)
\dot{S}	Entropy production rate, W/K
t	Thickness between walls with single shield between, m^*
T	Absolute temperature, K
x	Distance from cold wall, m^*
x'	Distance from cold wall in a multi-shield configuration, m^*
X	x/t , dimensionless distance*
X'	x'/L , dimensionless distance*
γ	Defined by Eq. (8)

Subscripts

C	Cold wall
H	Hot wall
i	i-th shield
min	Minimum
opt	Optimum
S	Shield

*For systems with single shield $L = t$, $x = x'$, $X = X'$.

INTRODUCTION

The search for the ultimate, energy efficient insulation system has led in the past few years to a fascinating rediscovery and application of some fundamental concepts of thermodynamics: specifically, the second law and the use of entropy production rates and availability (or exergy) for design optimization purposes. The classical approach has been to minimize the heat flow between surfaces at different temperatures.

The concept of a single vapor-cooled shield in an insulation has been treated theoretically as far back as 1959 in Scott's classic textbook on cryogenics [1] and designs employing them were described not much later [2]. Paivanas, et al., obtained a patent [3] and later reported on the use of uniformly spaced multiple shields which were cooled by the boil-off from the insulated dewar [4]. Eyssa and Okasha [5] considered only radiative heat exchange between shields and minimized the total refrigeration power required. Hilal, et al., [6,7] used a similar minimization of refrigeration power as the design basis. Related works were reported by Bejan, et al., [8-11].

Recently, Bejan [12] proposed a new point of view, based on the second law of thermodynamics, which considers thermal insulations as dissipators of useful mechanical power (i.e. the availability or exergy) or, alternately, as generators of irreversibility or entropy. Thus, in this method, optimization of an insulation corresponds to minimization of either the entropy production rate or the irreversibility, or the decrease of availability. Various applications of this concept to insulation systems have been documented subsequently [13,14].

Our work grew out of an examination of Cunnington's paper [13] who utilized a numerical technique to find optimum temperatures at given locations for one and two shields for a thermal conductivity function of the form

$k_1 T^{0.6}$. Although several equations seemed to be incorrectly printed we have found two of the design curves to be essentially correct. Thus, our purpose was

1. To develop a simple optimization technique;
2. To generalize the results to a broader class of insulations; and
3. To develop simple design methods for cooled shields.

The essentials of this report were already published [15].

ANALYSIS

We accept the previously developed concept that to optimize an insulation system is equivalent to minimizing the entropy production rate. In addition, we assume one-dimensional heat flow and that the heat capacity of the boil-off gas is adequate to do the cooling for all shields and does not impose a restriction on the optimization. In contrast to Rejan [9,11] who has developed a constrained optimization based on the heat capacity of the boiloff we employ the argument that in all practical systems the boil-off is generated by cooling of some equipment in addition to the heat leakage across the insulation.

Parallel heat paths, e.g. supports, have not been considered. However, each path can be optimized separately using its own thermal conductivity function. Then a design decision has to be made whether the two structures should be independently cooled at their respective optimum conditions.

We examine the general situation of an insulation where equivalent thermal conductivity, k , can be expressed as a two-term function of the absolute temperature

$$k = k_1 T^m + k_2 T^n \quad (1)$$

where, typically, the first term represents actual conduction with $m \geq 1$ and the second term represents radiation with $n \geq 3$. In the following, m and n can be any value except -1.

The heat flow across a layer of insulation can be expressed in terms of Fourier's law

$$q \, dx = Ak \, dT \quad (2)$$

Substituting k from Eq. (1) and integrating across a layer from one end at 1, to the other at 2, yields

$$q = \frac{A}{x_2 - x_1} \left[\frac{k_1}{m+1} (T_2^{m+1} - T_1^{m+1}) + \frac{k_2}{n+1} (T_2^{n+1} - T_1^{n+1}) \right]. \quad (3)$$

Now consider the insulation with a cooled shield at T_S located at x between a hot surface at T_H and a cold one at T_C , separated by the insulation thickness, t , as shown in Fig. 1a. The entropy production rate for the insulation can be determined from the heat flows and temperatures as follows

$$\dot{S} = -\frac{q_H}{T_H} + \frac{q_C}{T_C} + \frac{q_S}{T_S} \quad (4)$$

where $q_S = q_H - q_C$.

The heat flow terms can be expressed in the form of Eq. (3) and the resulting expression can be non-dimensionalized using the following terms

$$s \equiv \frac{\dot{S}t}{Ak_H} \text{ where } k_H = k \text{ at } T_H, \quad (5)$$

$$P \equiv \frac{T_S}{T_C}, \quad (6)$$

$$R \equiv \frac{T_C}{T_H}, \quad (7)$$

$$Y \equiv \frac{k_2(m+1)}{k_1(n+1)} T_H^{n-m}, \quad (8)$$

and

$$X \equiv \frac{x}{t}. \quad (9)$$

The resulting equation is

$$\begin{aligned}
 & s(m+1) \left(1 + \gamma \frac{n+1}{m+1}\right) \\
 &= \frac{1}{1-X} \{[(PR)^{m+1} - (PR)^m - 1 + (PR)^{-1}]\} \\
 &+ \gamma [(PR)^{n+1} - (PR)^n - 1 + (PR)^{-1}] \\
 &+ \frac{1}{X} \{R^m [P^{m+1} - P^m - 1 + P^{-1}]\} \\
 &+ \gamma R^n [P^{n+1} - P^n - 1 + P^{-1}] \quad (10)
 \end{aligned}$$

Since R, the overall temperature ratio, is generally known, s is a function of P and X, and its extreme value can be found by differentiating it with respect to each variable separately and setting the results equal to zero. This procedure yields two equations to be solved simultaneously: $\partial s / \partial P = 0$ and $\partial s / \partial X = 0$. Because of the regular form of the expressions, one of the final two equations contains only a single unknown as follows:

$$\begin{aligned}
 & \frac{R^m F(m,P) + \gamma R^n F(n,P)}{[R^{m-1} D(m,P) + \gamma R^{n-1} D(n,P)]^2} \\
 &= \frac{F(m,PR) + \gamma F(n,PR)}{[D(m,PR) + \gamma D(n,PR)]^2} \quad (11)
 \end{aligned}$$

$$\frac{X}{1-X} = - \frac{R^{m-1} D(m,P) + \gamma R^{n-1} D(n,P)}{D(m,PR) + \gamma D(n,PR)} \quad (12)$$

where the following functionals were used:

$$F(b,B) \equiv R^{b+1} - B^b - 1 + B^{-1} \quad (13)$$

$$D(b,B) \equiv (b+1) B^b - bB^{b-1} - B^{-2}. \quad (14)$$

Thus, to find the optimum temperature and location for a shield, Eq. (11) can be solved for P , and then X can be calculated from Eq. (12). The heat to be removed by the shield, $q_S = q_H - q_C$, can be found, as before, from Eq. (3). In dimensionless form the equation becomes

$$\begin{aligned} \frac{q_S t}{Ak_H T_H} (m+1) \left(1 + \gamma \frac{n+1}{m+1}\right) \\ = \frac{1 - (PR)^{m+1} + \gamma[1 - (PR)^{n+1}]}{1 - X} \\ = \frac{(PR)^{m+1} - R^{m+1} + \gamma[(PR)^{n+1} - R^{n+1}]}{X}. \end{aligned} \quad (15)$$

For multiple shields t_i represents the distance between the two surfaces surrounding the i -th shield on either side, $T_{H,i}$ and $T_{C,i}$ are the temperatures of these two surfaces, $X_i = x_i/t_i$ is the location of the shield relative to t_i , and x_i' is the location of the shield relative to the cold wall as shown in Fig. 1b. To determine the optimum temperatures and locations for multiple shields, first we assumed a temperature for the first shield next to the cold wall, then we used Eqs. (11) and (12) to find the temperature and location of the second shield. This process was repeated for the rest of the shields and the hot wall. Thus, each shield was optimized consecutively with respect to the two surfaces on either side. With given values of the overall temperature ratio, R , and of the number of shields, the process requires iterative solution.

To put the results into proper perspective, the entropy production rates can be compared to the thermodynamically minimum rate obtainable through spatially continuous cooling. According to Bejan [12], this rate is

$$\dot{S}_{\min} = \frac{A_c}{t} \left[\int_{T_C}^{T_H} (k)^{1/2} T^{-1} dT \right]^2. \quad (16)$$

This expression was evaluated analytically for the single-term functions of k , i.e. for $\gamma = 0$, and numerically otherwise.

RESULTS AND DISCUSSION

The first set of curves, Figs 2 through 9, show the relative entropy production rates for various thermal conductivity functions and for up to four optimally cooled shields as functions of the overall temperature ratio $R \equiv T_C/T_H$. The curves show that the entropy production rate increases with decreasing values of the temperature ratio, R , and with increasing values of the exponent, m and n . Adding shields, of course, reduces the entropy production rate; but for most of the practical temperature range, say $0.01 < R < 0.4$, only three shields contribute to significant decreases and adding a fourth shield can be considered unnecessary. No shields are useful at high values of R ; but this "high" range is strongly dependent on the exponent of the temperature. The curves developed with $k = k_1 T^{0.6}$ for one and two shields were very close to those given by Cunnington [13], converted appropriately.

Study of the results of two-term conductivities reveals that the curves fall between those obtained for each of the two terms alone. If γ is small the first term, T^m , dominates; whereas if γ is large (>10), the second term, T^n , controls. Thus, general conclusions can be drawn from examining the results of the single-term conductivities.

The second set of curves, Figs. 10 through 31, show the optimum temperature ratios, T_S/T_H , and optimum locations, x'/L , of cooled shields as functions of the overall temperature ratio, T_C/T_H , for various thermal conductivity functions and with different number of cooled shields.

Figures 10 and 11 show the optimum single shield temperature ratios, $PR = T_S/T_H$, and locations, $X = x/L$, for five conductivity functions. Both of these functions generally decrease with decreasing R . The other figures in this set show shield temperatures and locations for systems with up to three

shields and for both single-term and two-term conductivities. The results are strongly non-linear. For example, for $k_1 T^3$ and $R = 0.01$, the optimum temperature ratios for three shields are about 0.09, 0.3, and 0.6 and the optimum locations are about 0.05, 0.2, and 0.5. As is to be expected, our unconstrained optimization yields a somewhat better performance per shield than Bejan's [9,11] constrained method.

The sensitivities of the entropy production rates to deviations from the optimum values of PR and X are demonstrated in the last set of curves, Figs. 32 through 35, for single shields. The sensitivity increases with the value of the exponents, m and n, but the curves are relatively flat near the minima. A ± 20 percent change from optimum, for example, has negligible effect. Thus, the system is relatively tolerant of deviations from the optimum design conditions.

Calculations with two different conductivities on the two sides of a cooled shield show that using the better insulator on both sides always yields the optimum condition. However, if for some reason two types of insulations have to be used, then the better insulator should be placed on the warm side of the shield.

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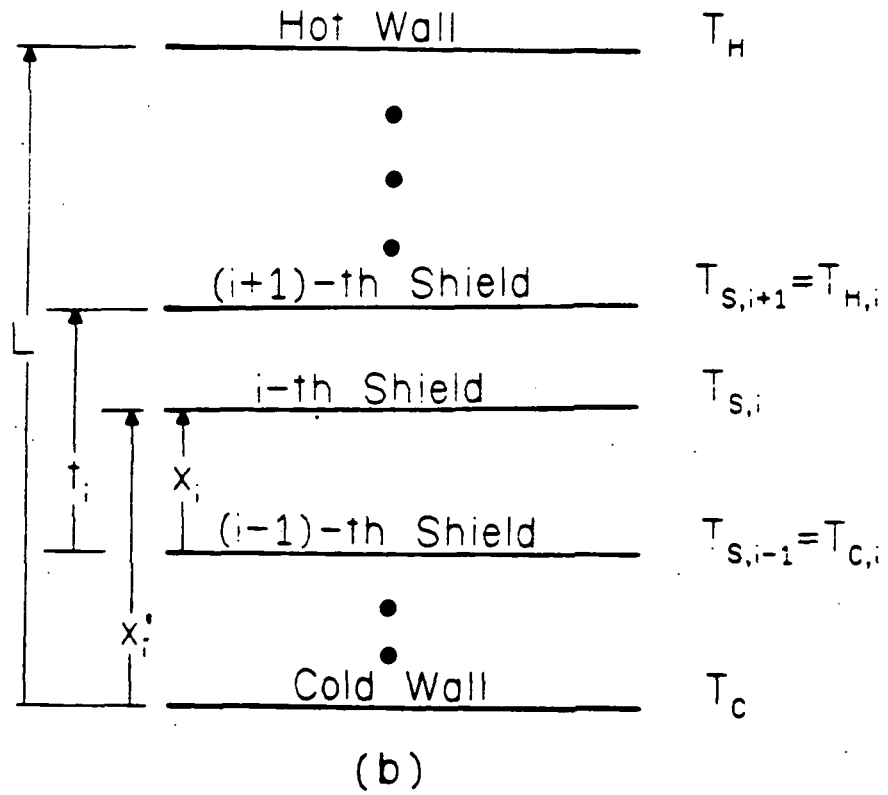
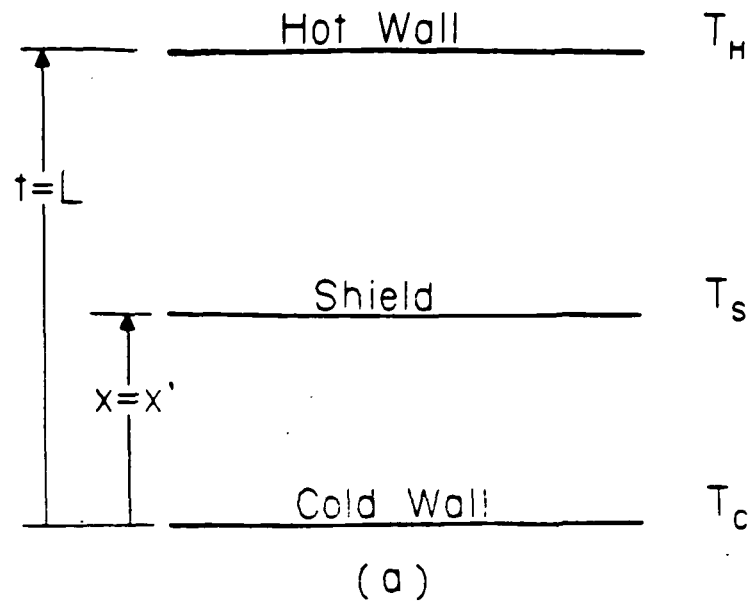


Figure 1 Schematic of the Nomenclature for (a) Single and (b) Multiple Shields

Curve Set 1: Figures 2 through 9

The effect of optimally cooled shields on
the entropy production rate for various thermal conductivities.

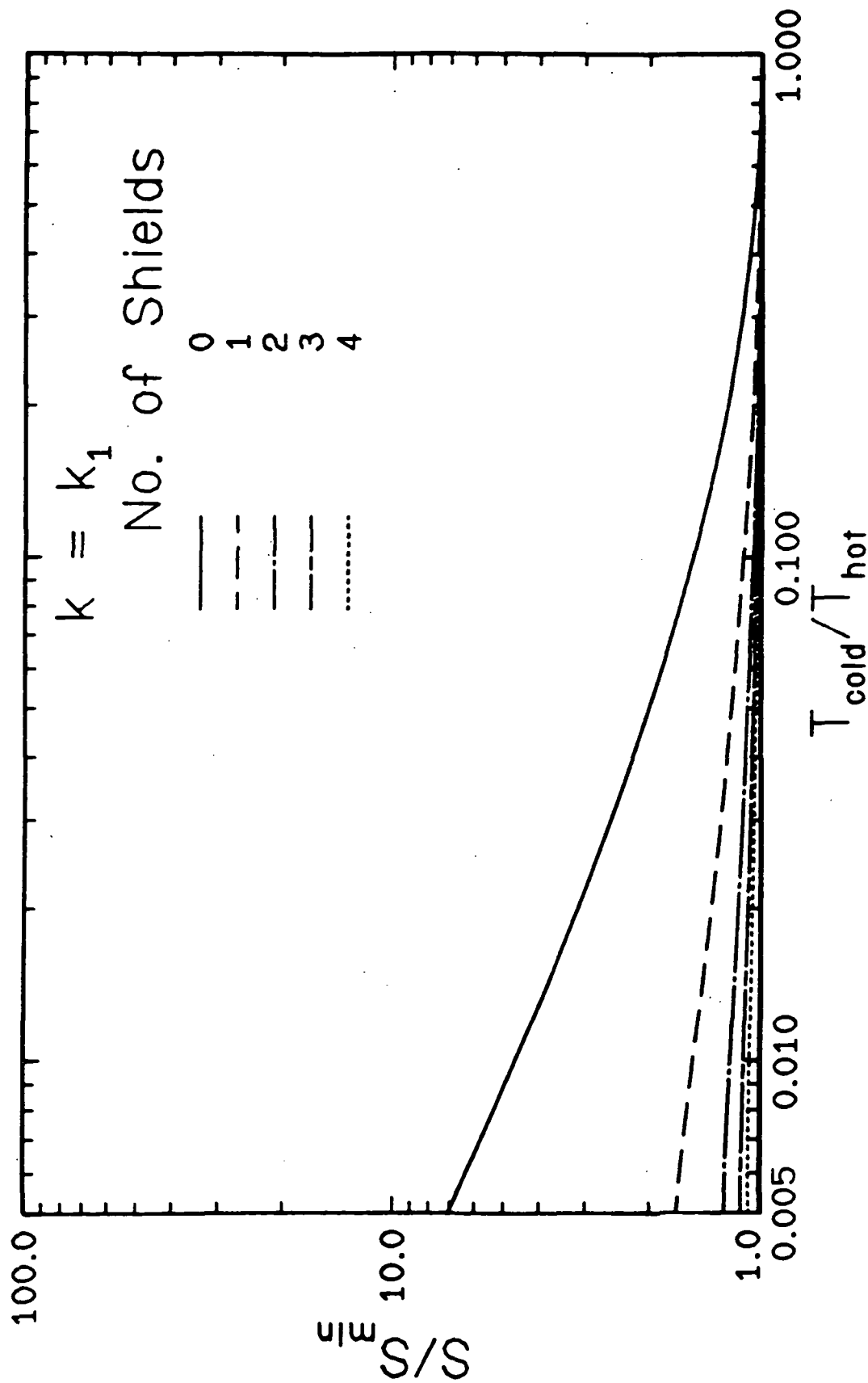


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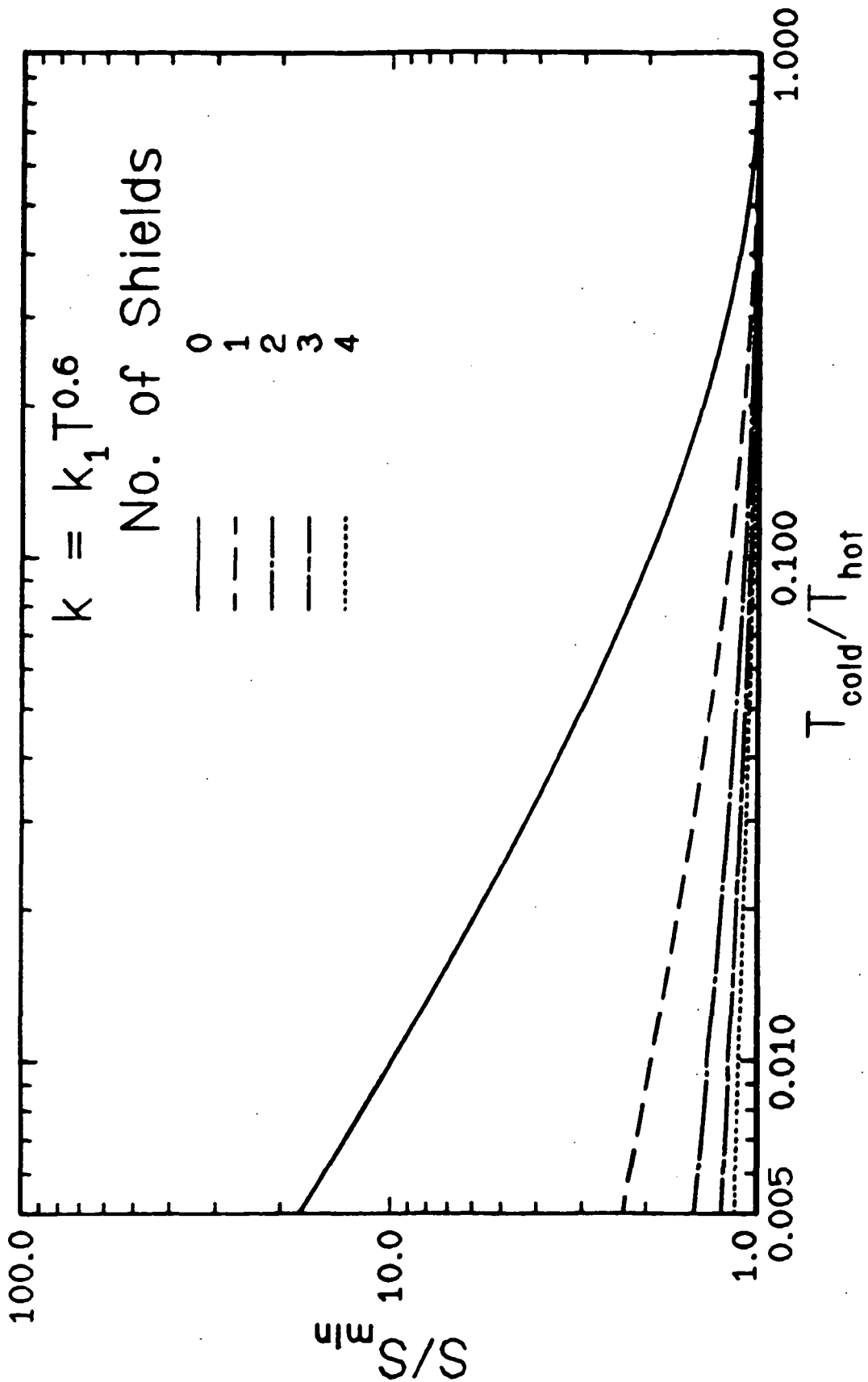


Figure 3

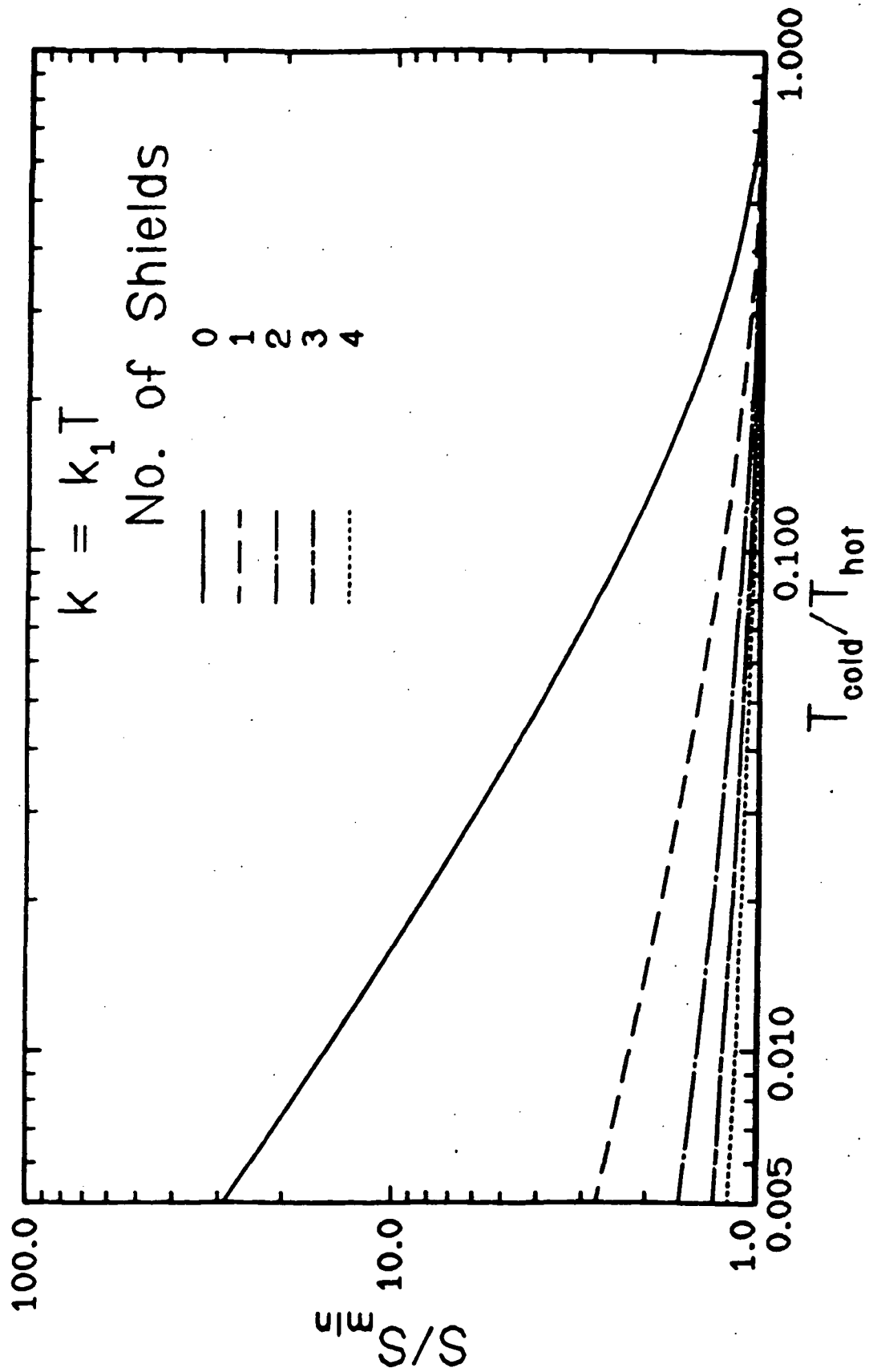


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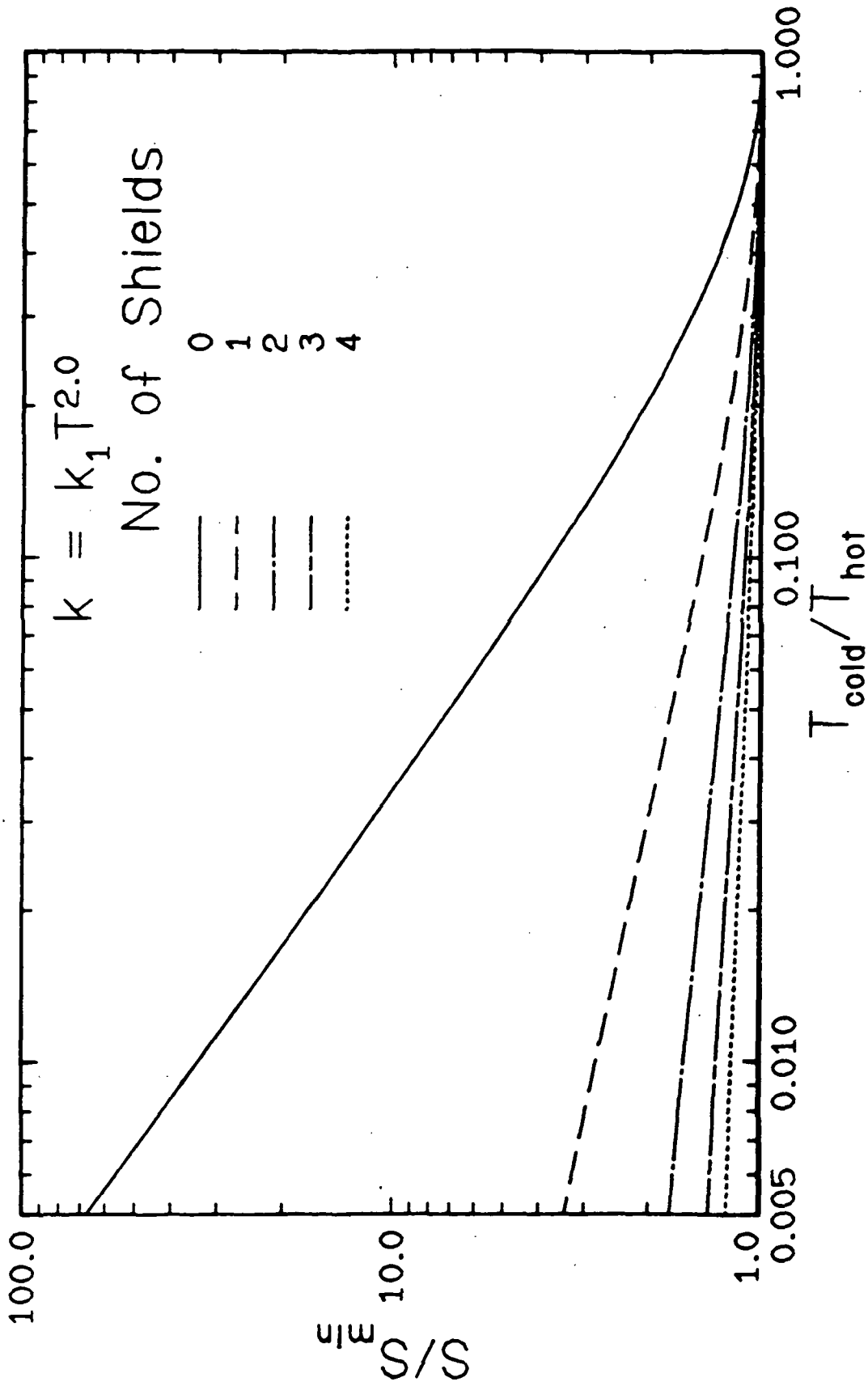


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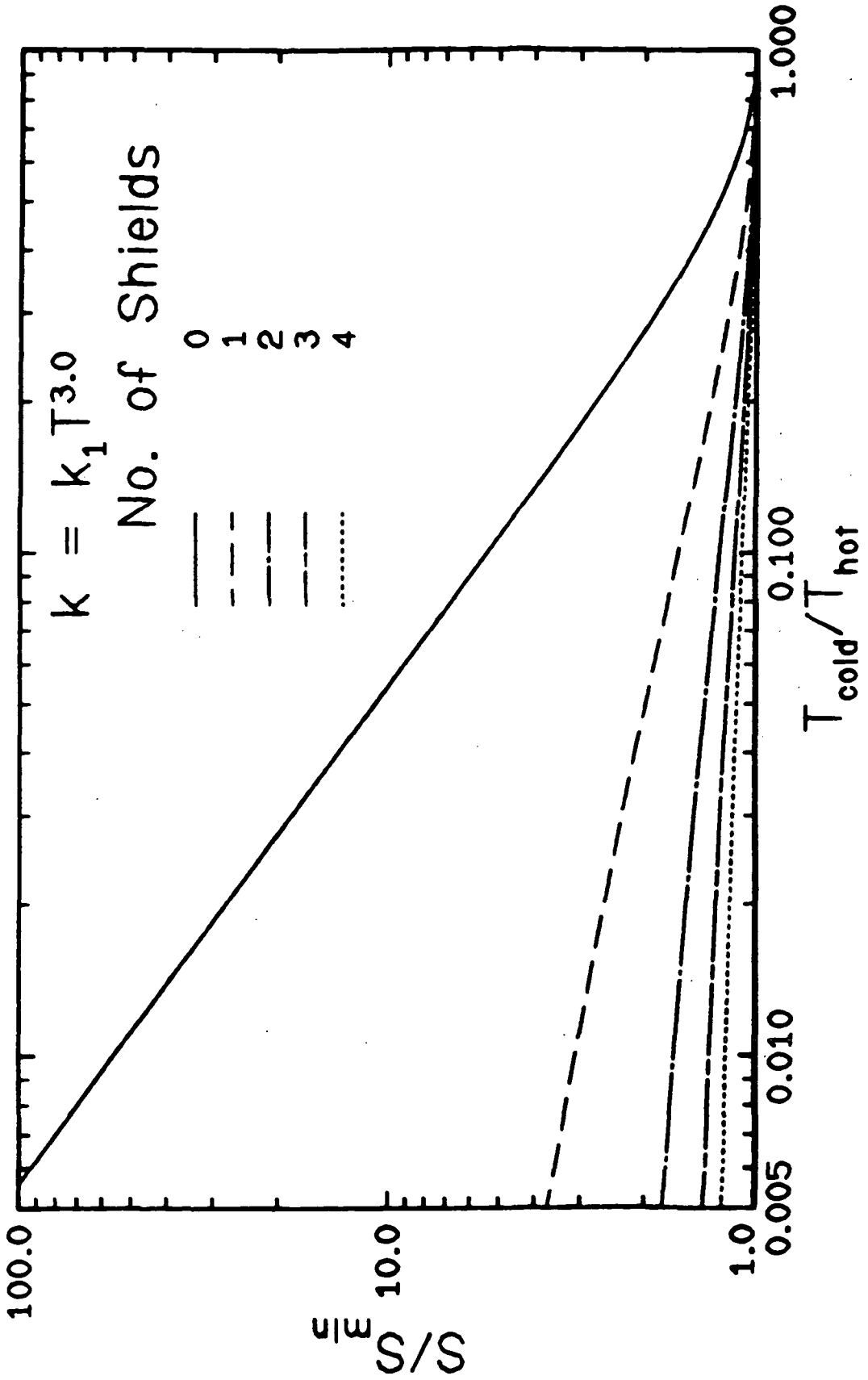


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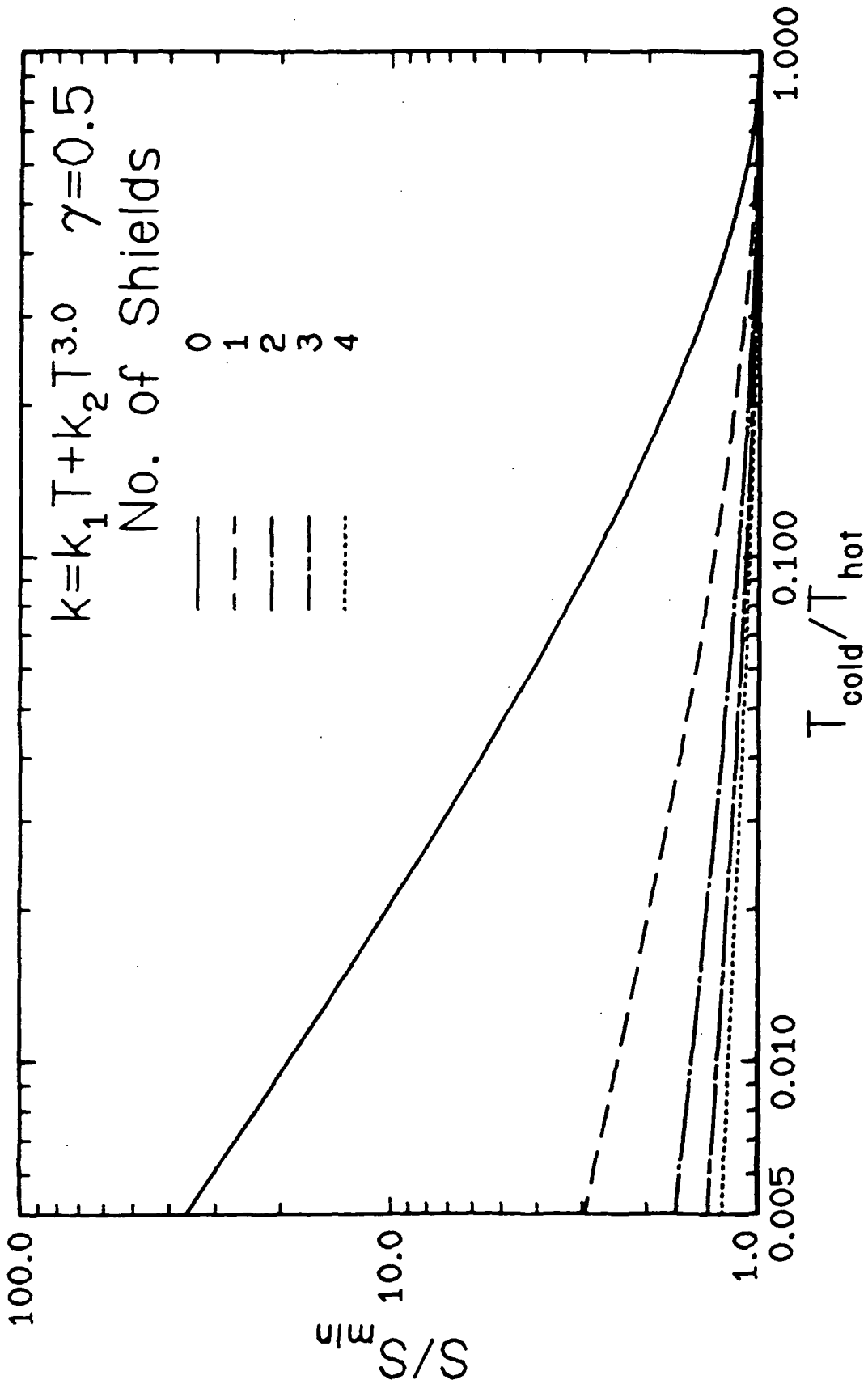


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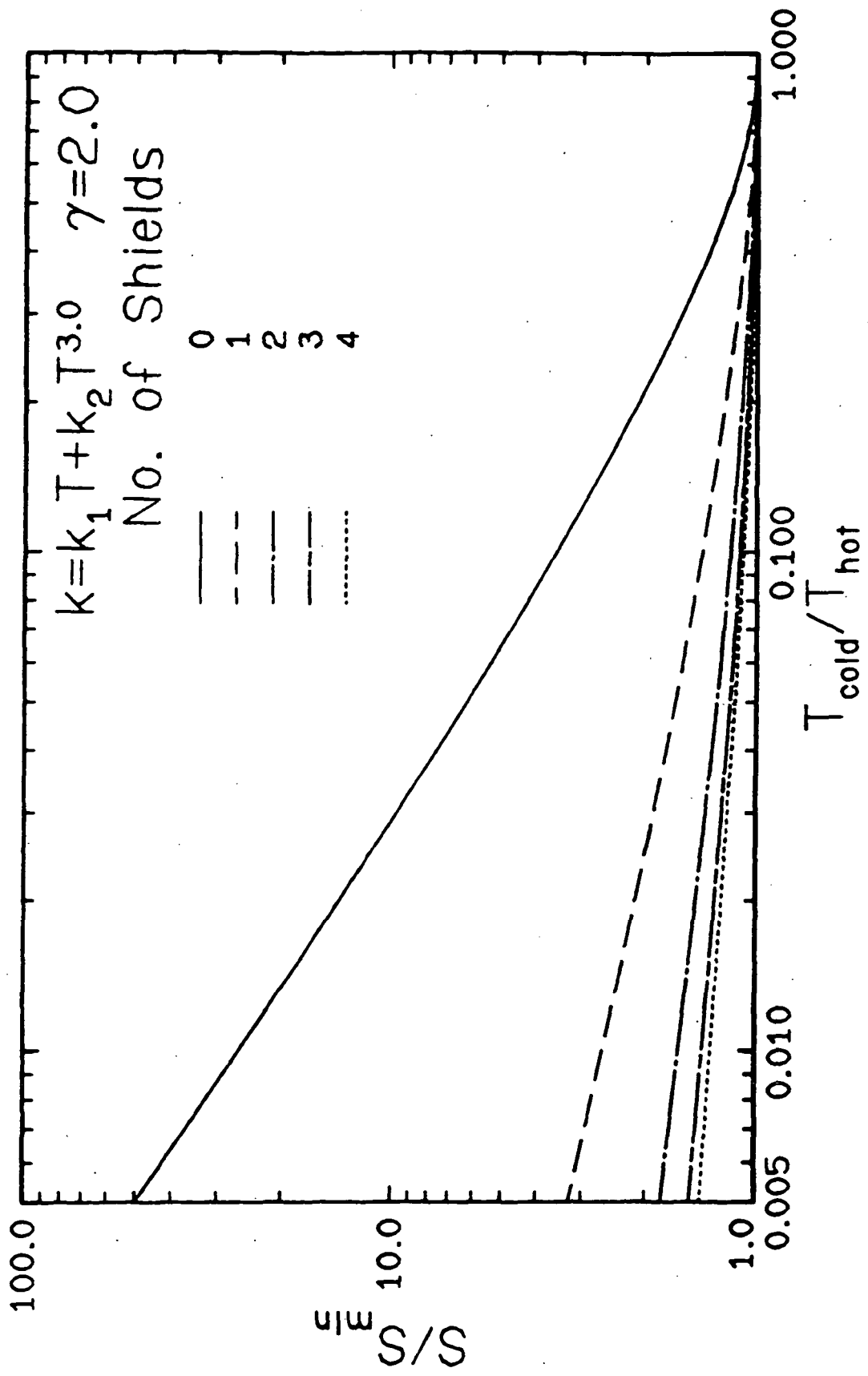


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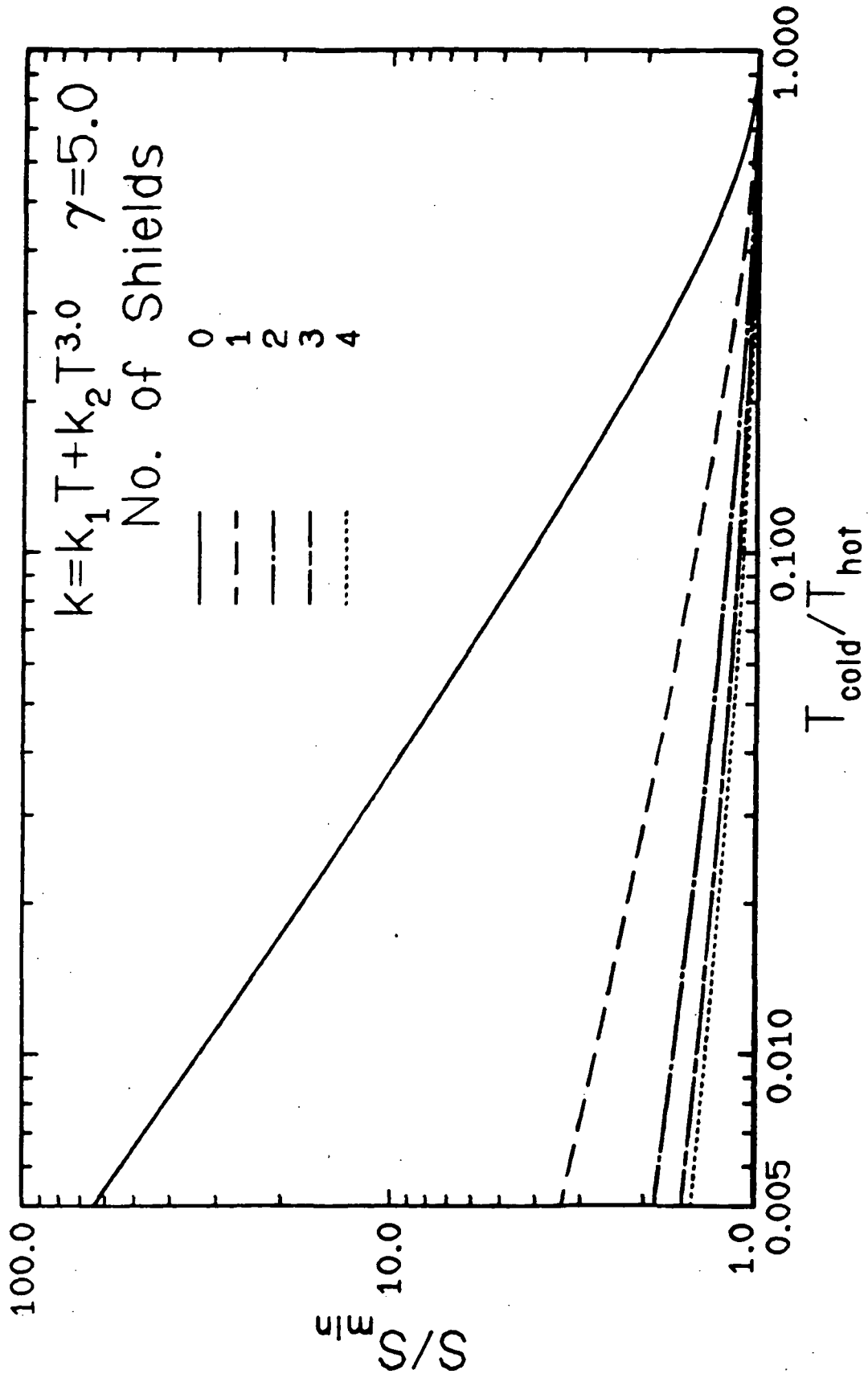


Figure 9

Curve Set 2: Figures 10 through 31

Optimal shield temperatures and locations for various thermal conductivity functions with different number of shields.

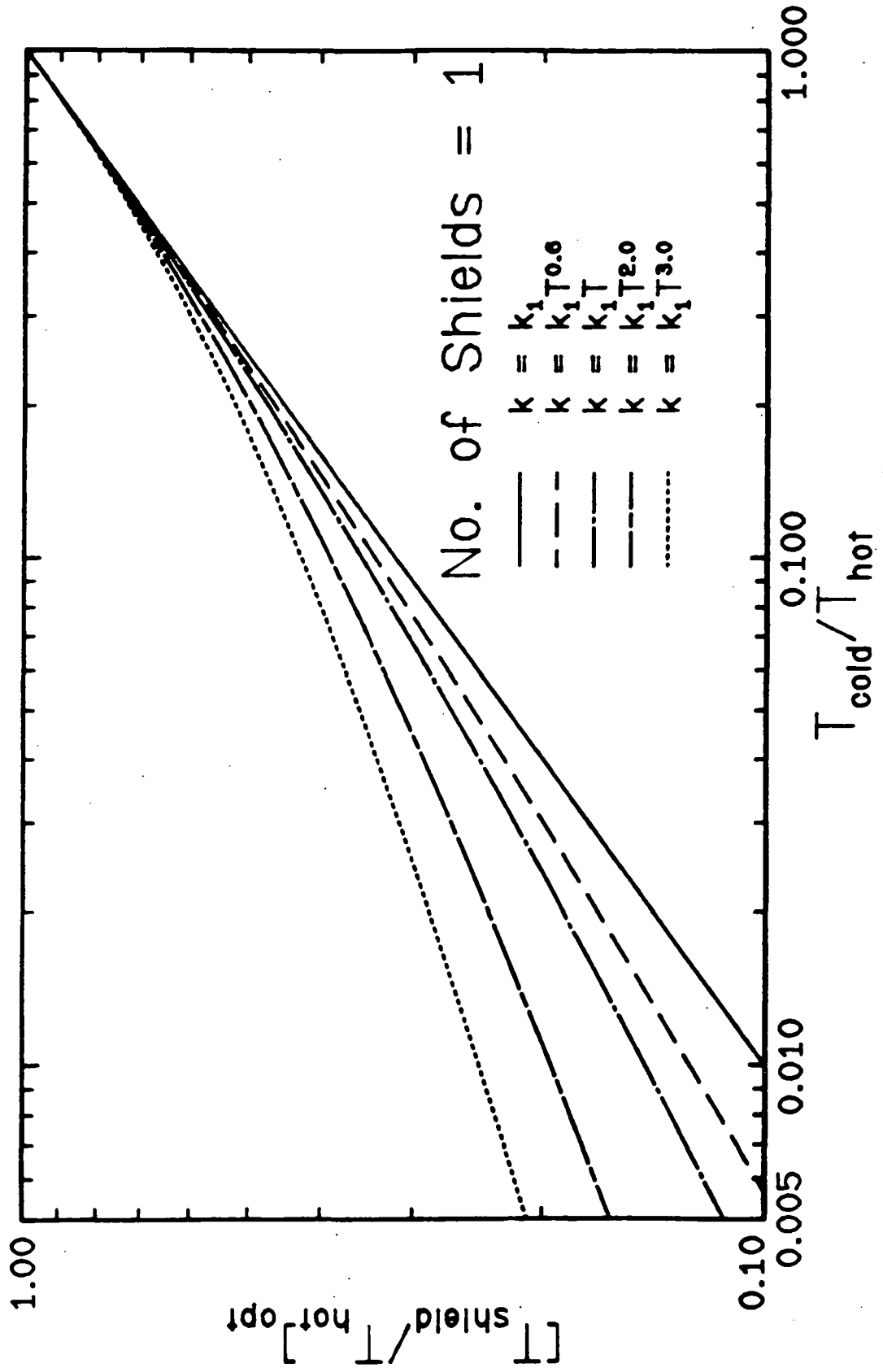


Figure 10

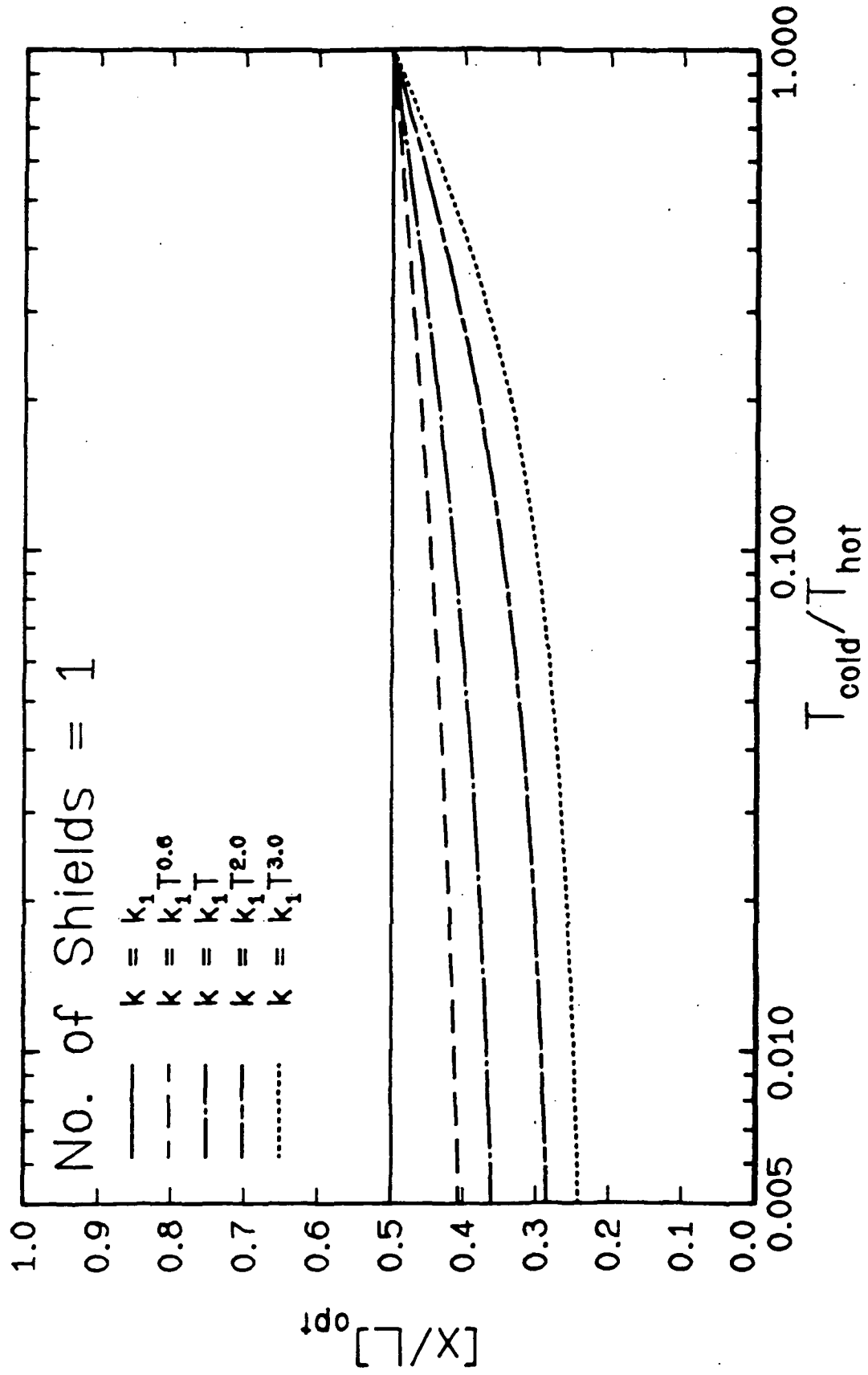


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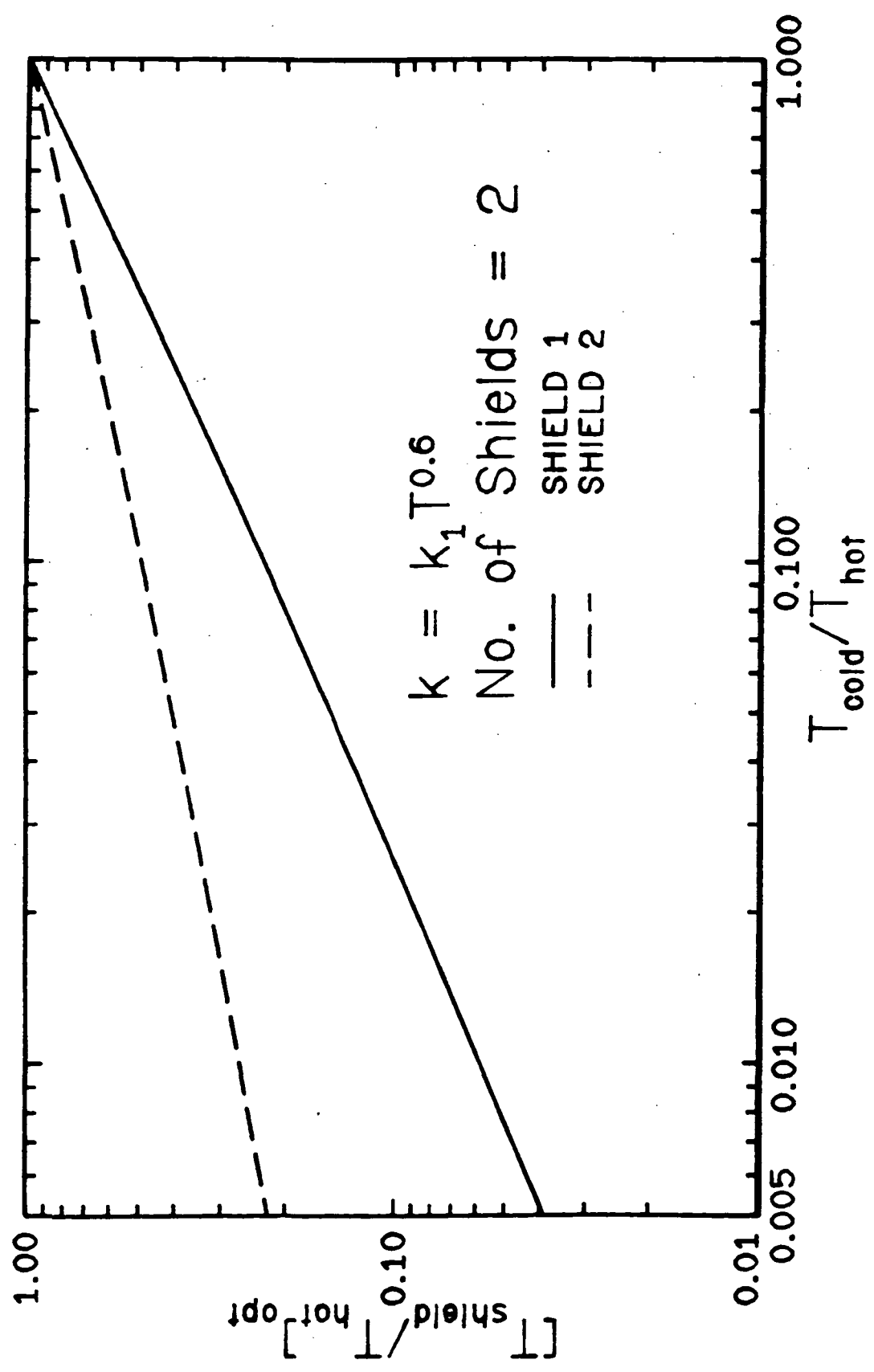


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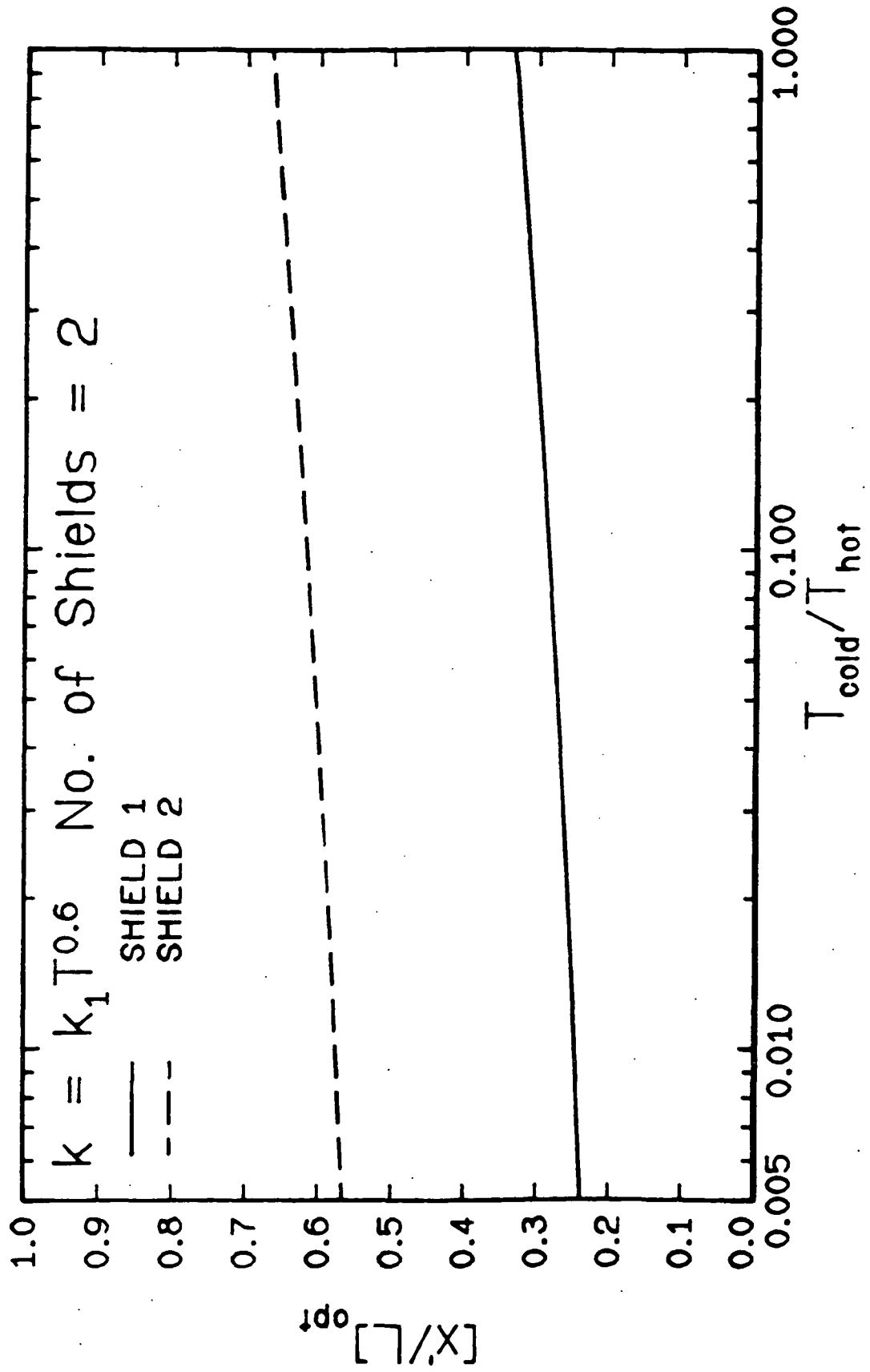


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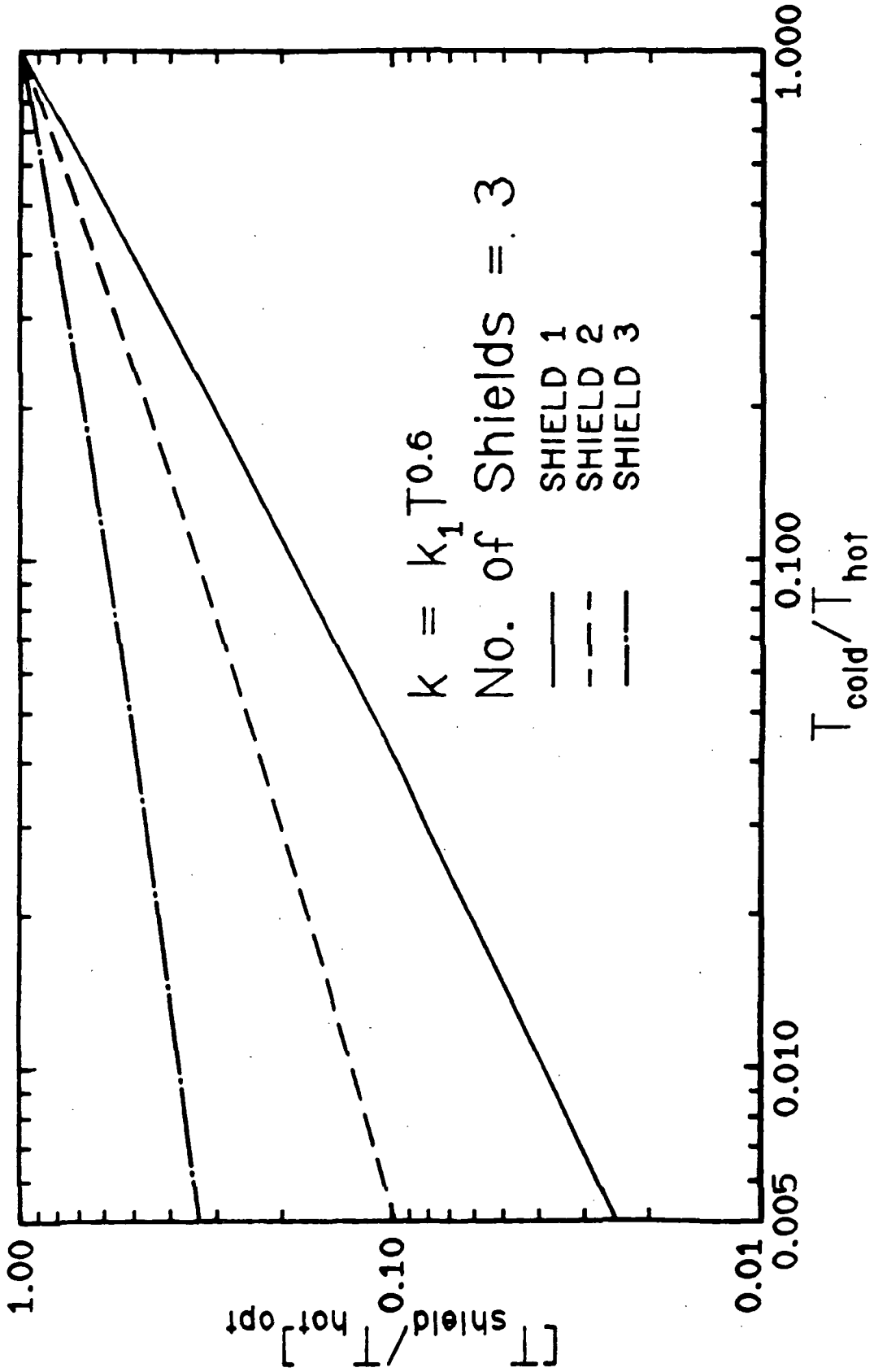


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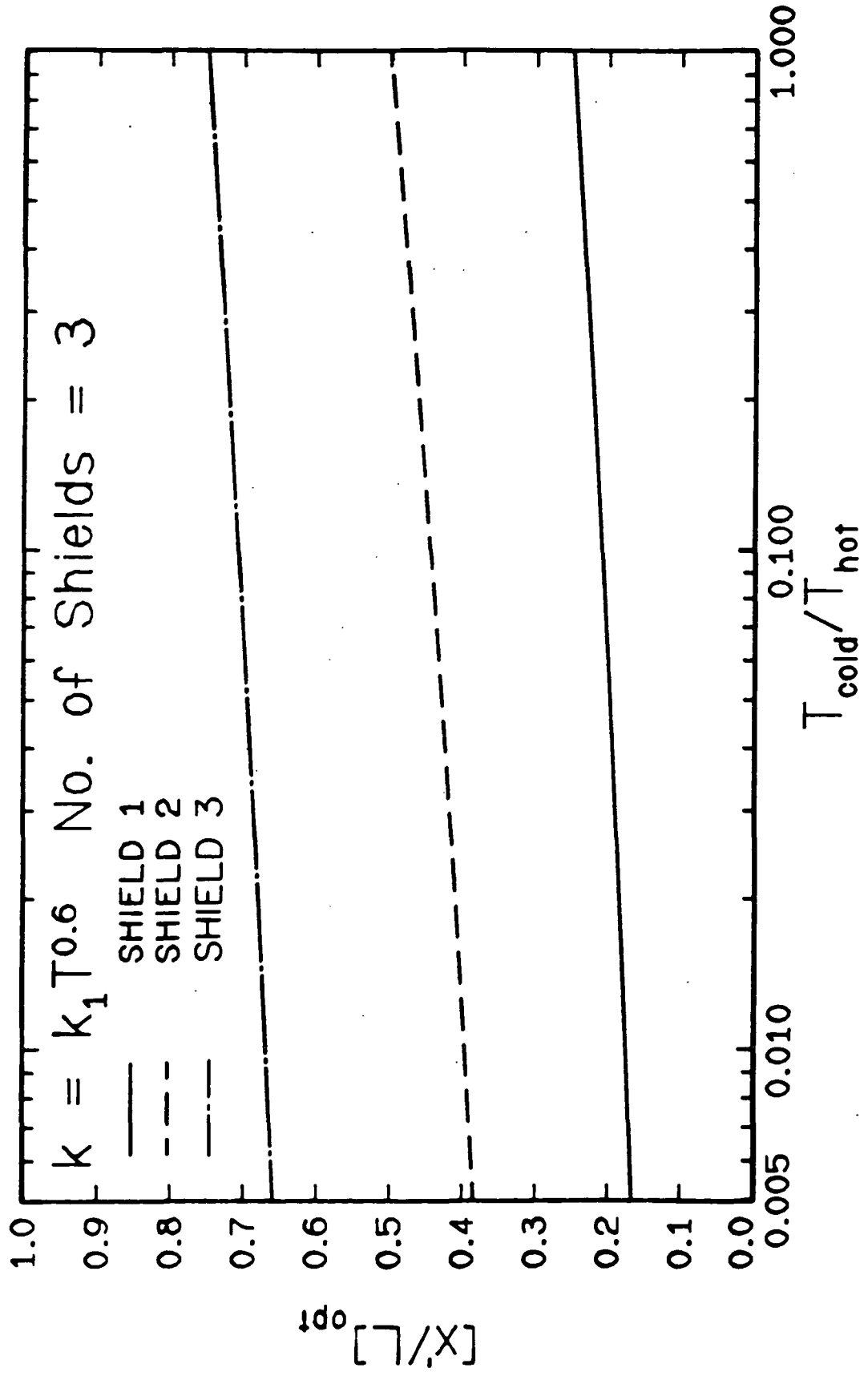


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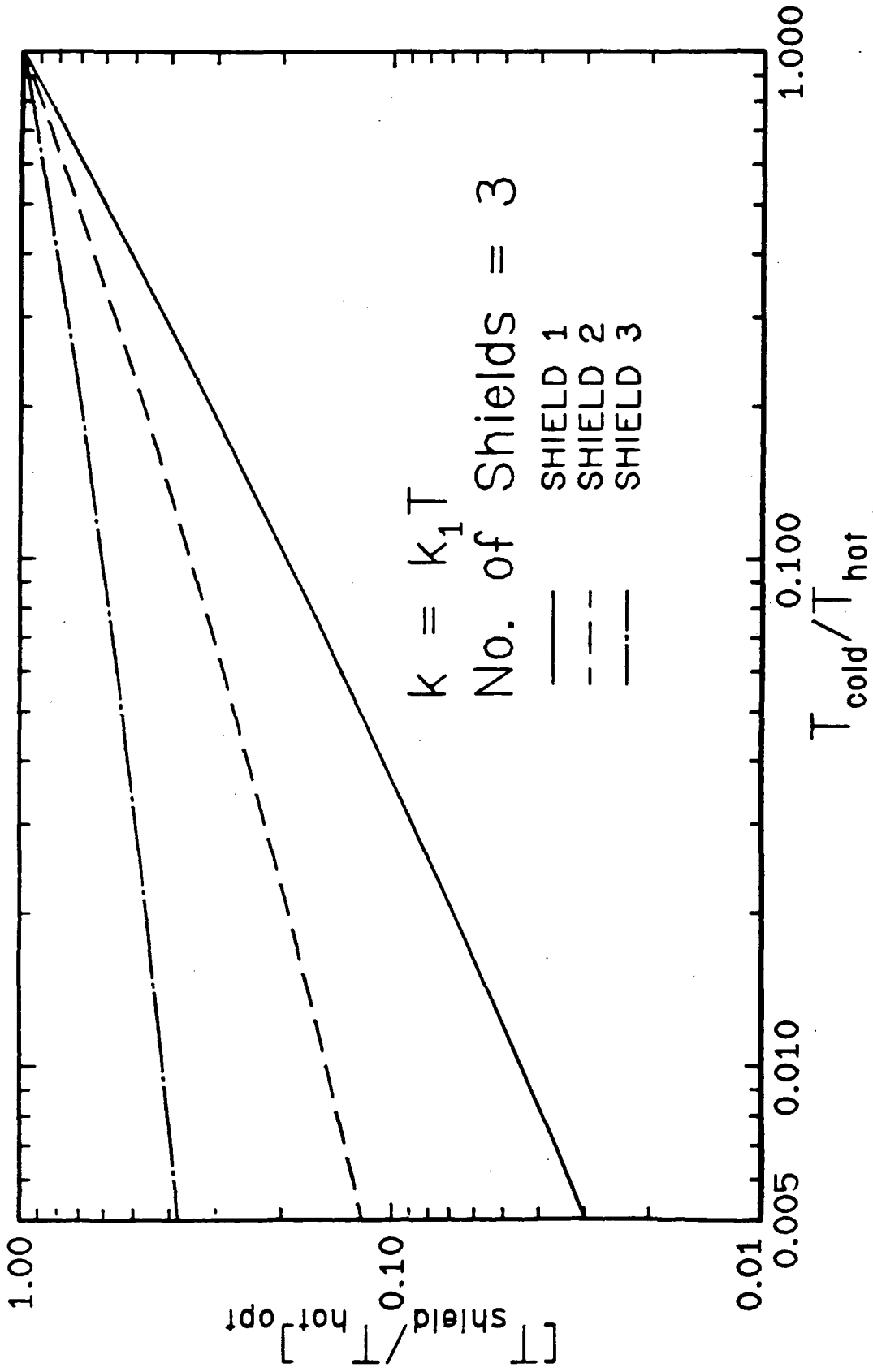


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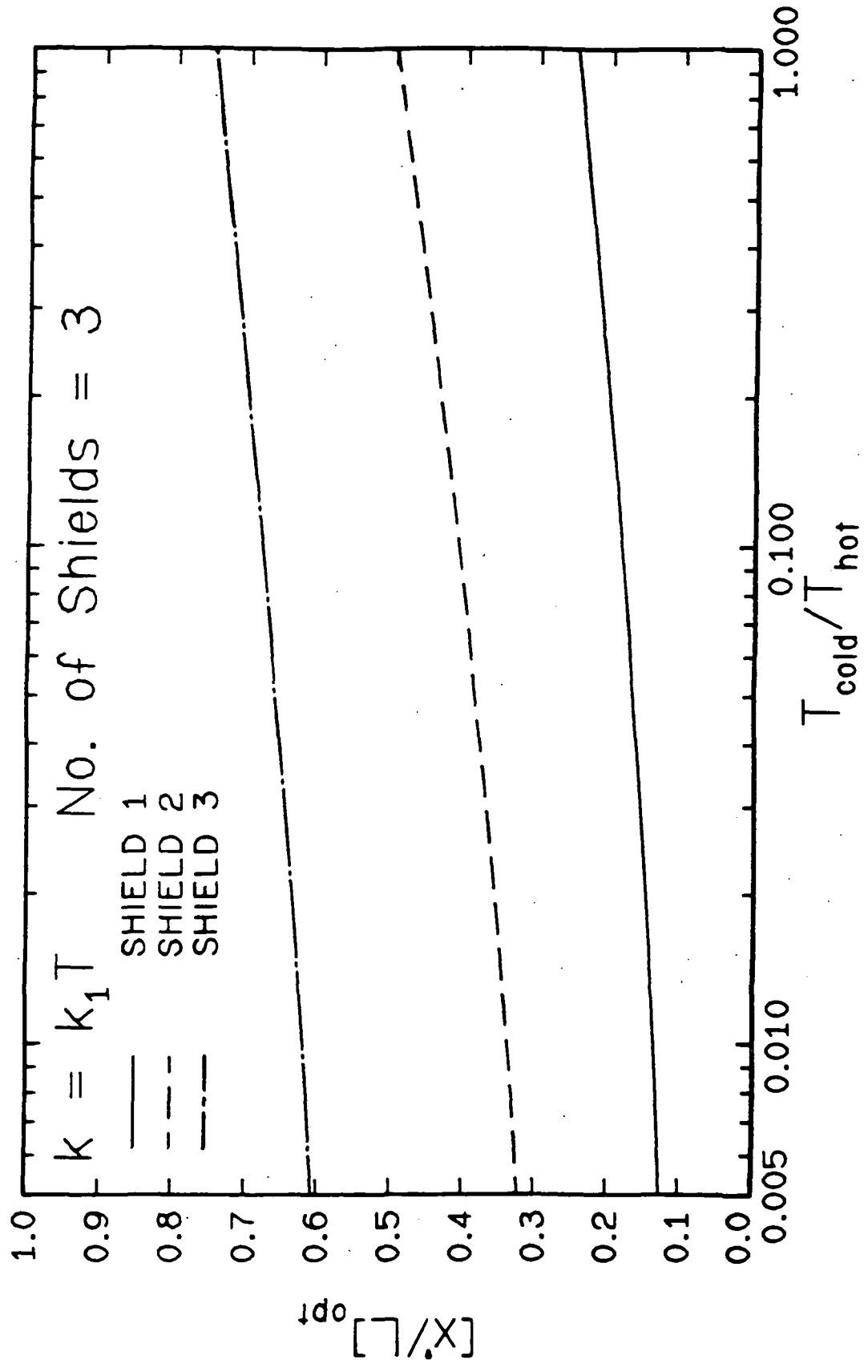


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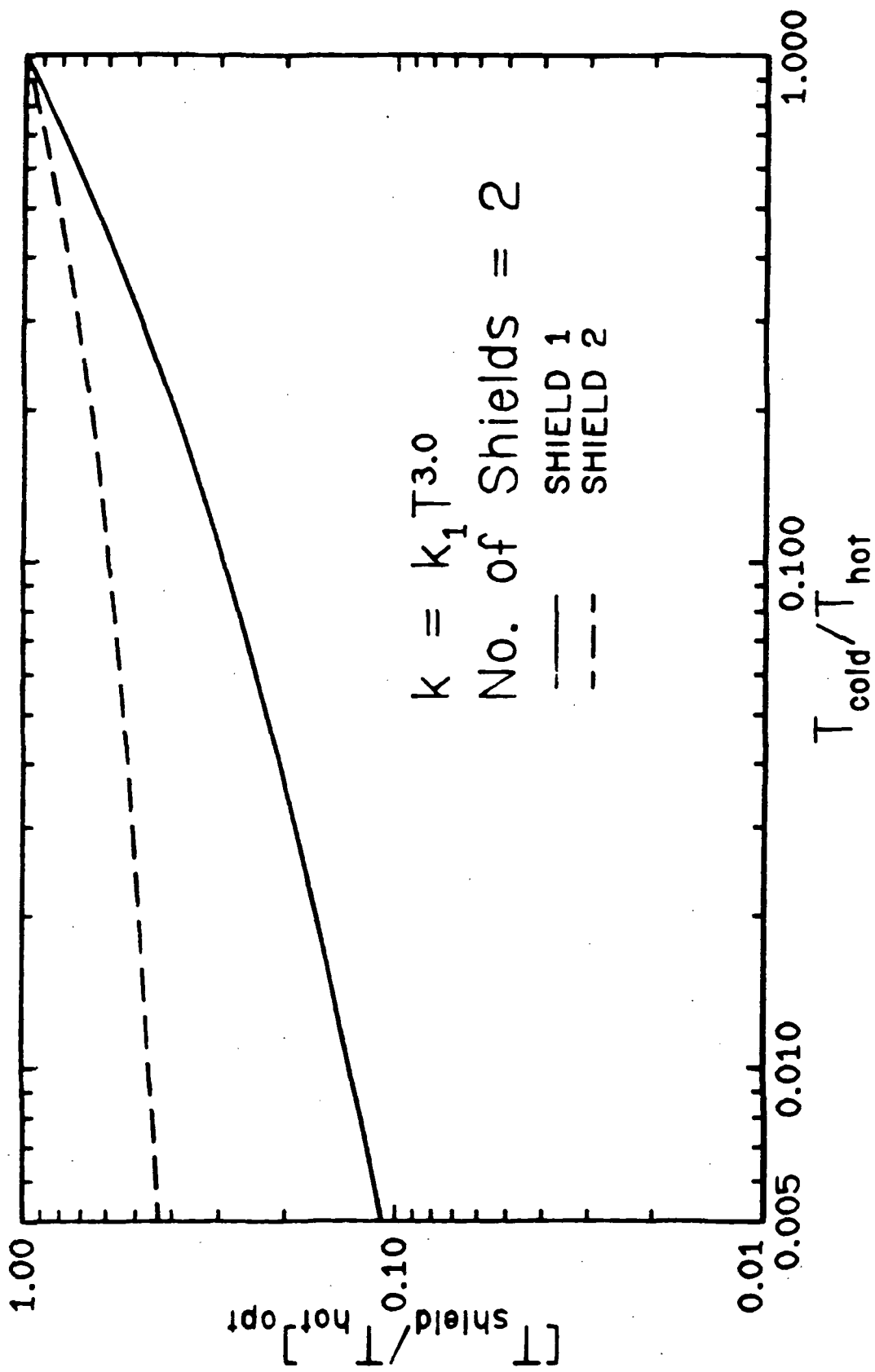


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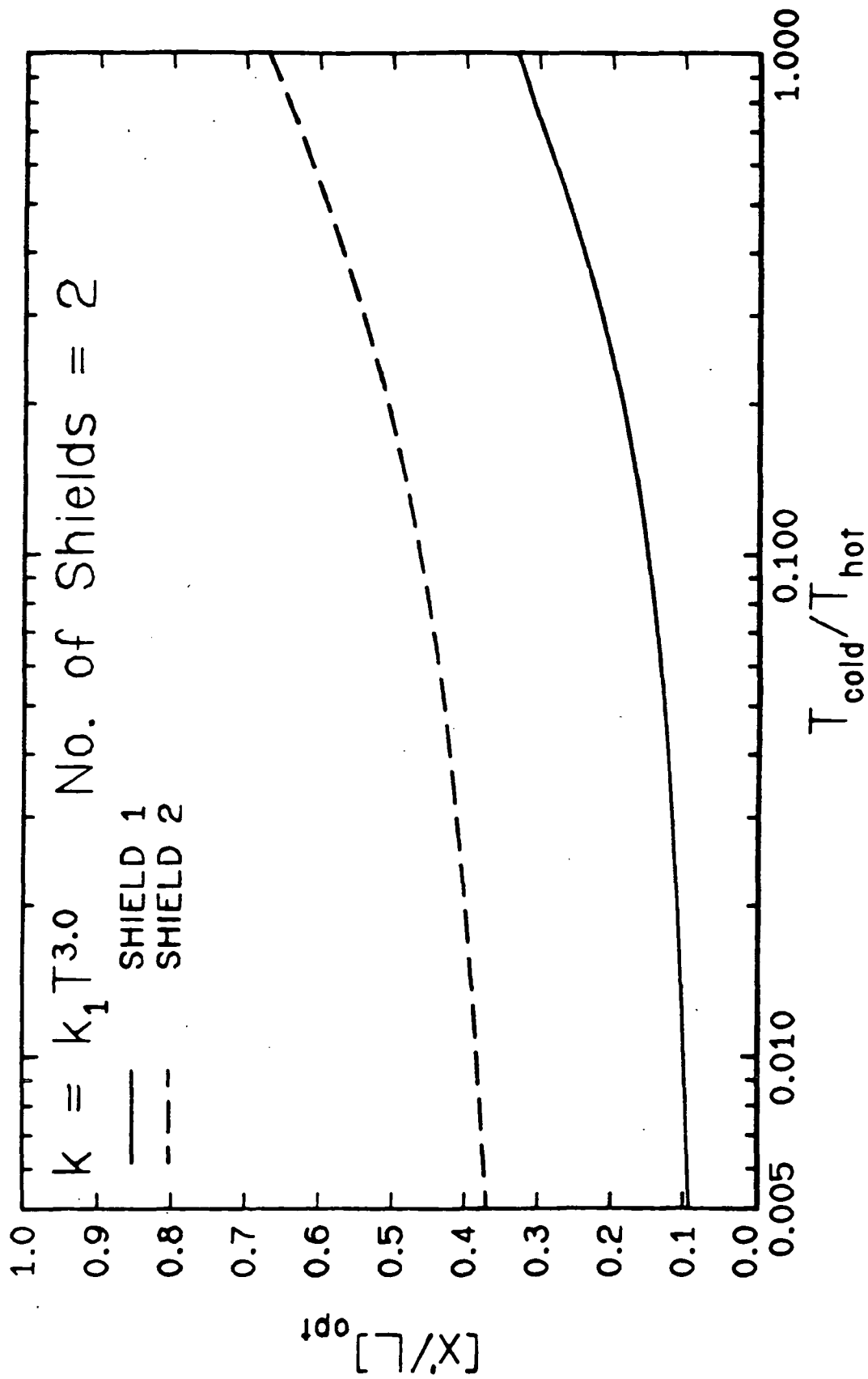


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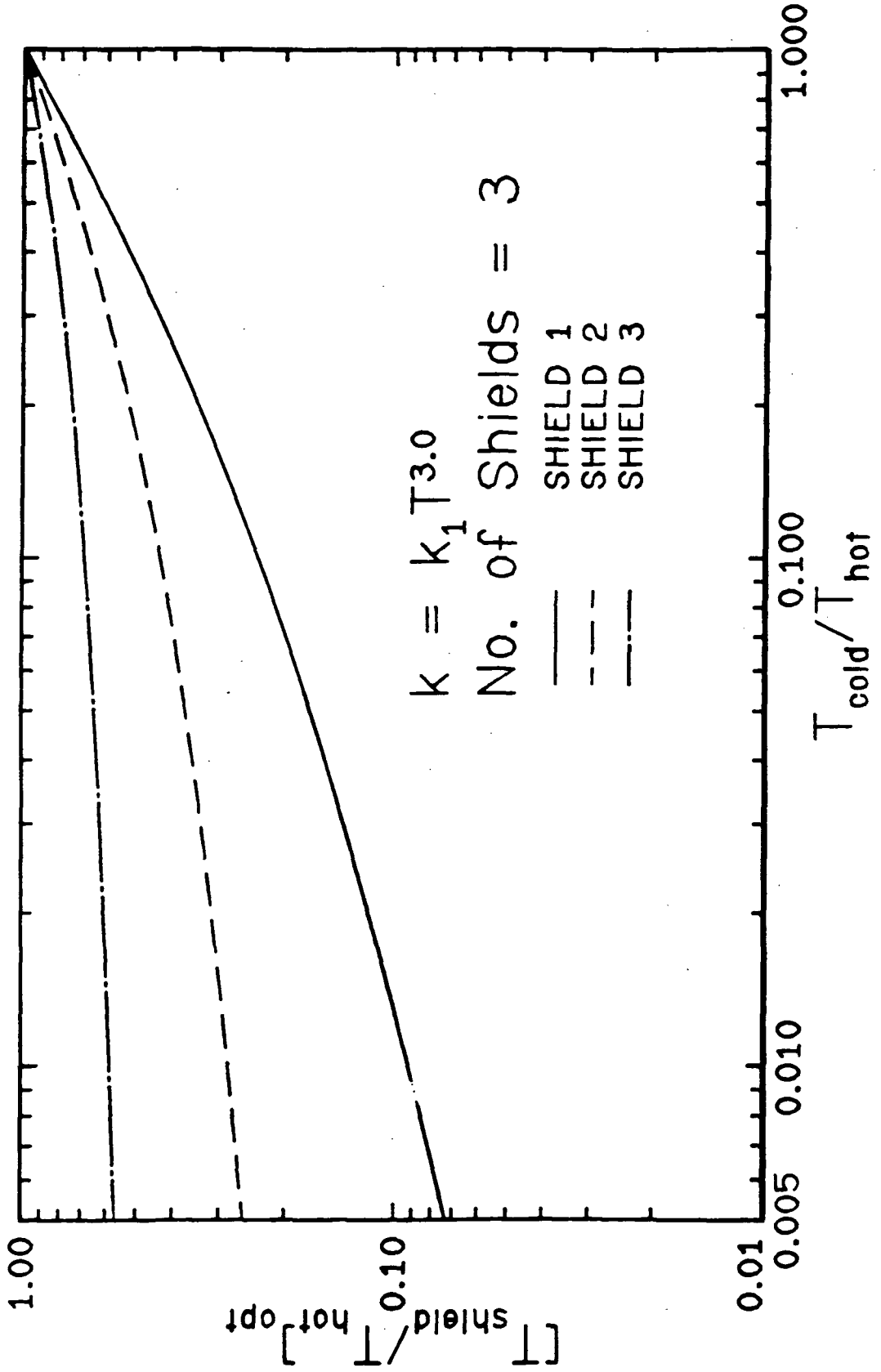


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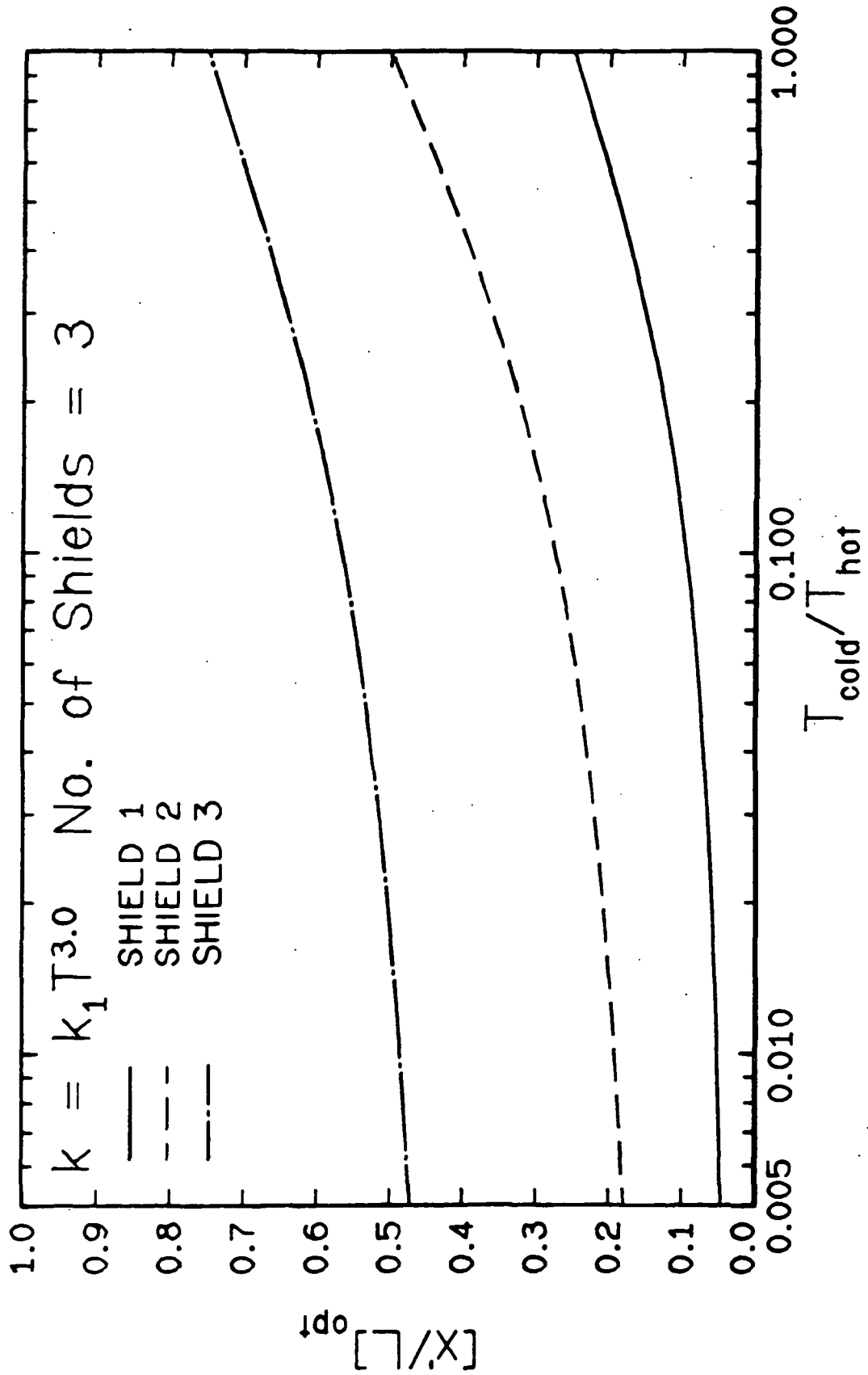


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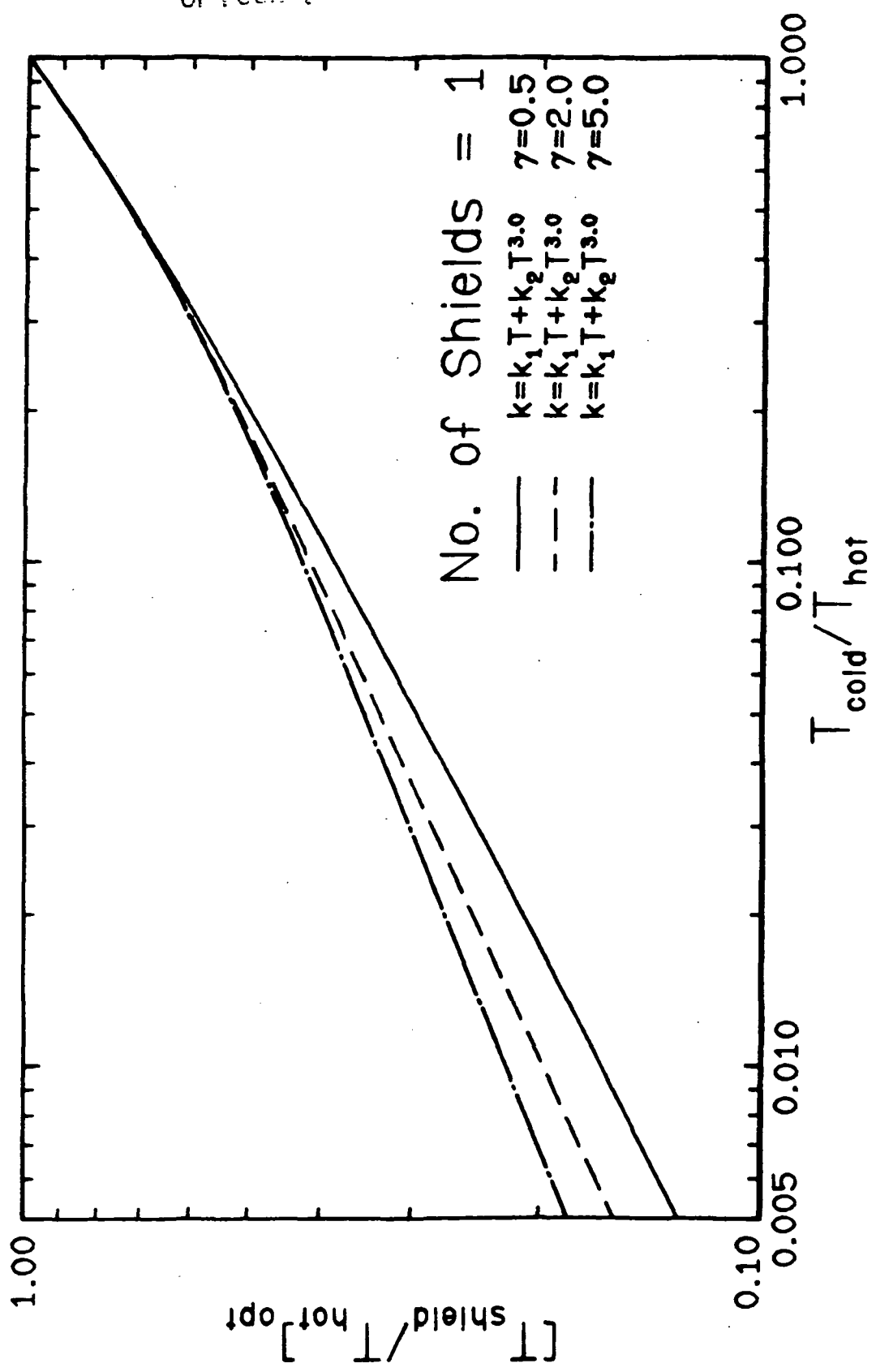


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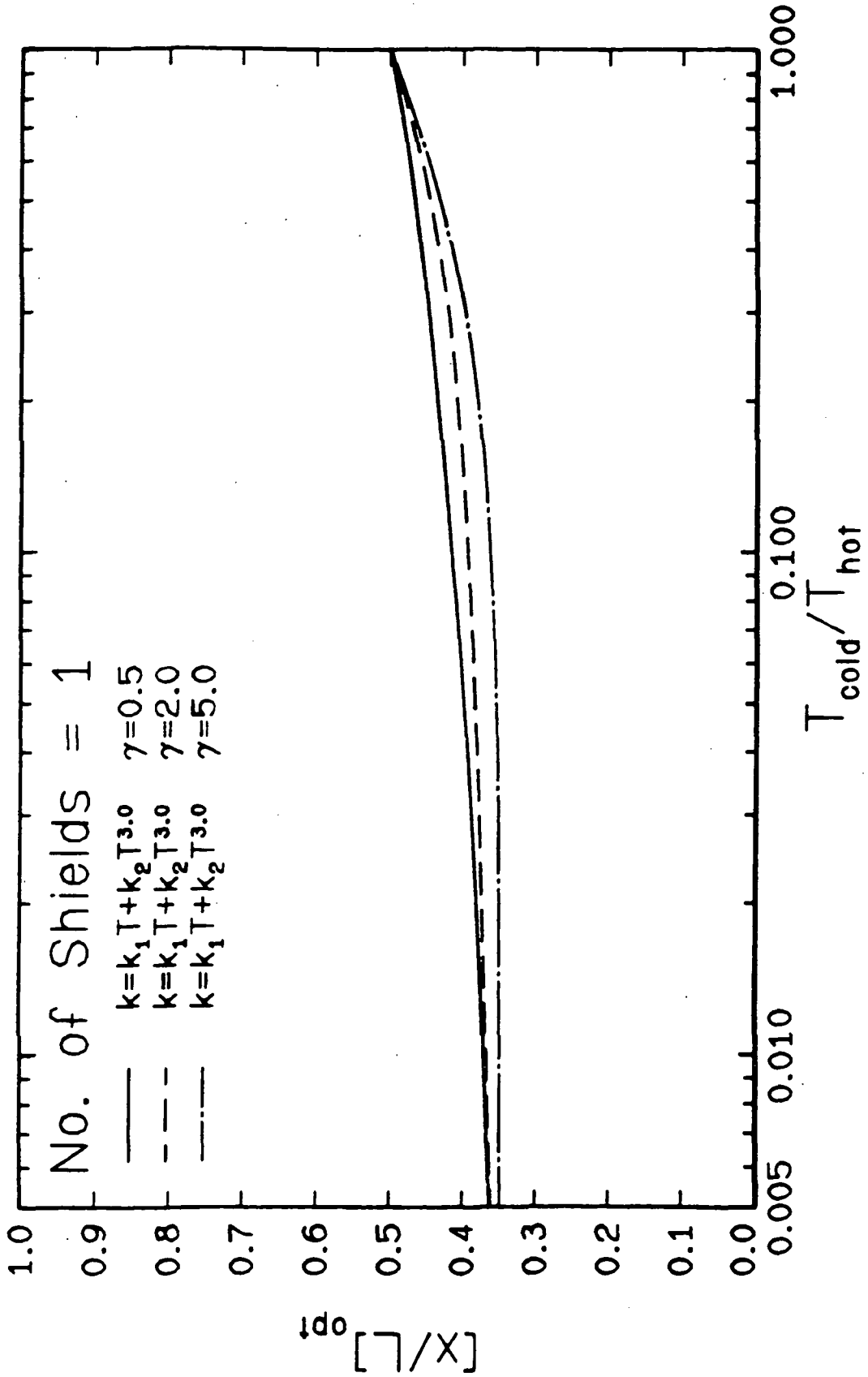


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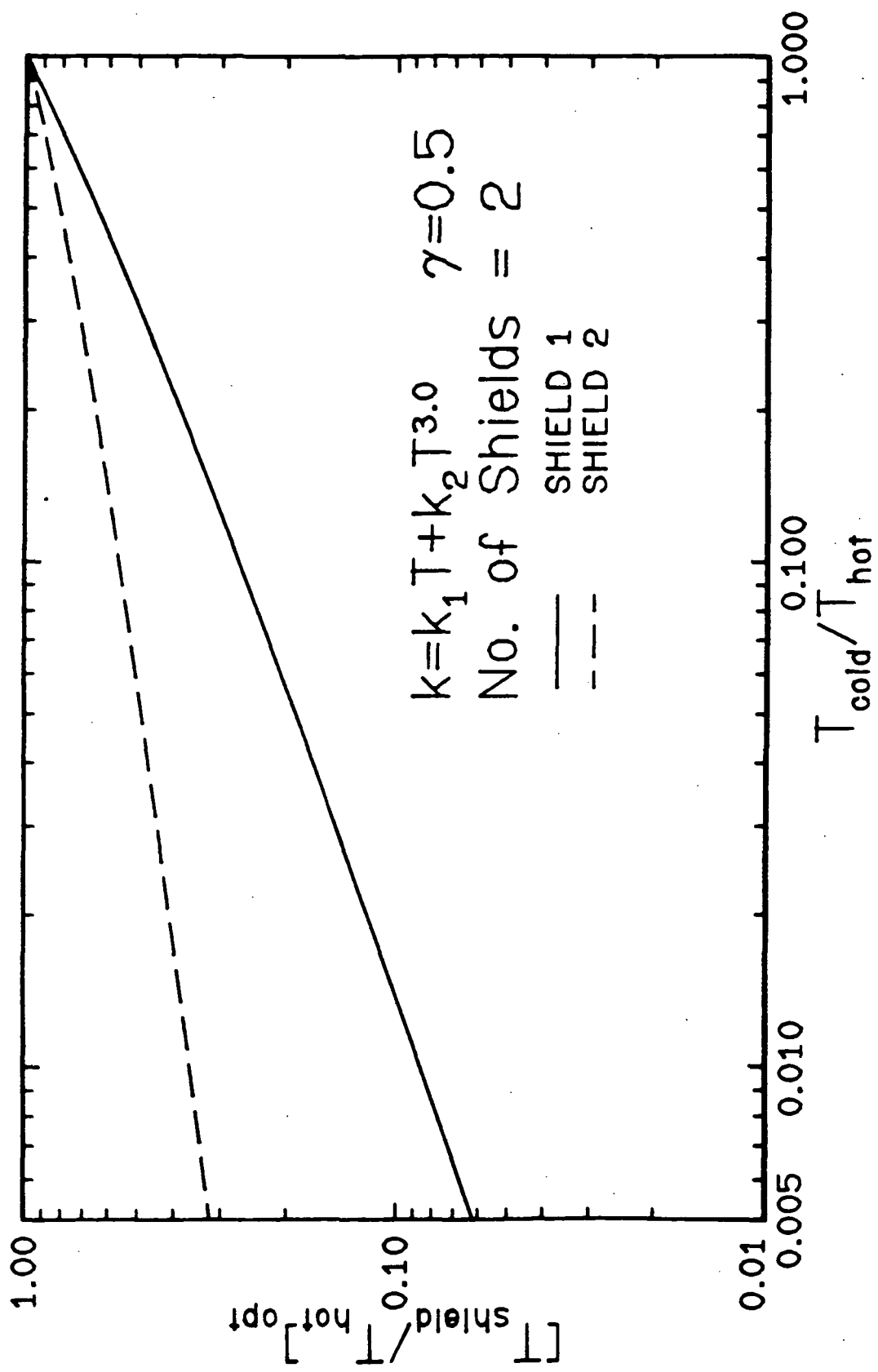


Figure 24

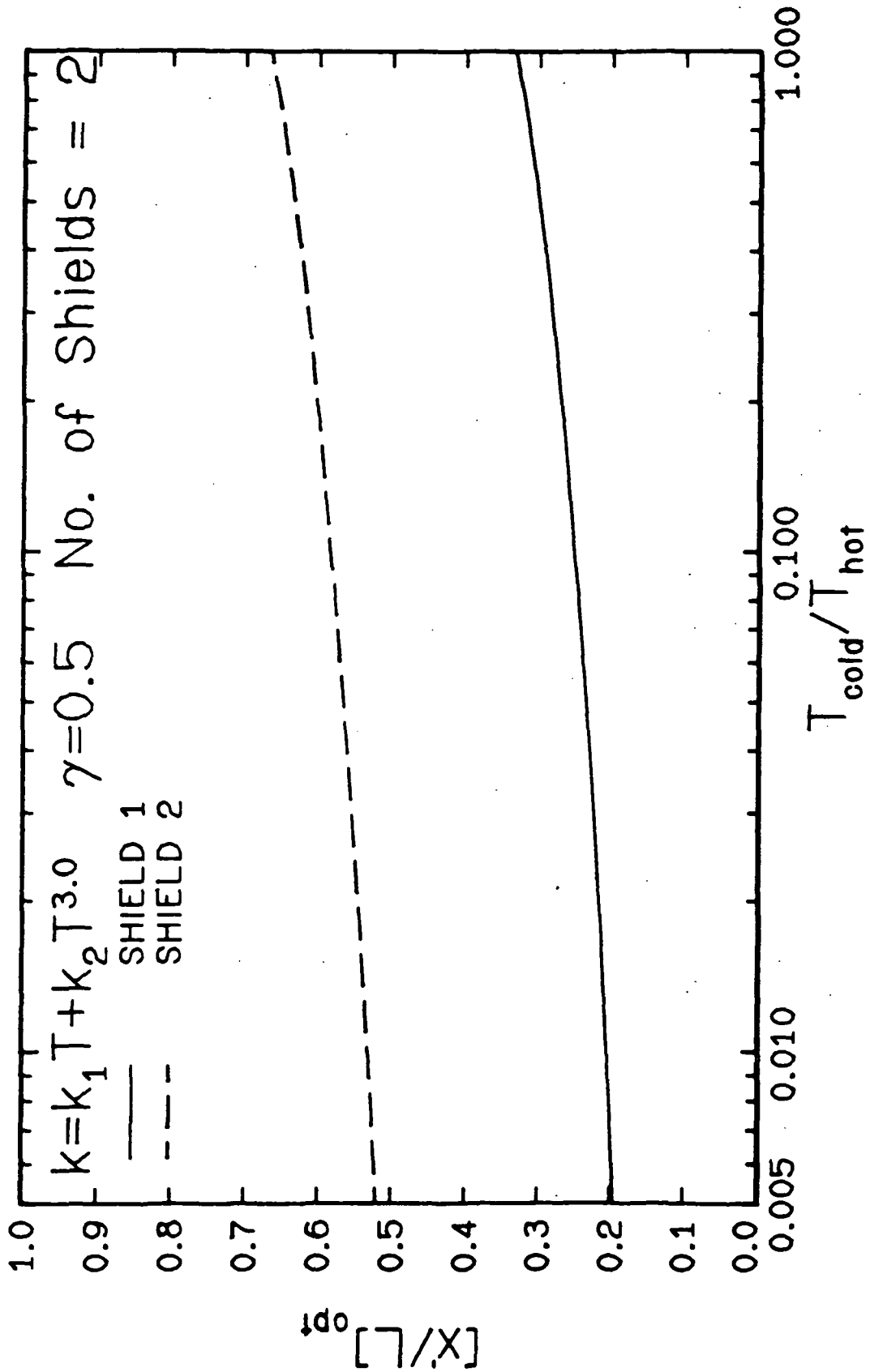


Figure 25

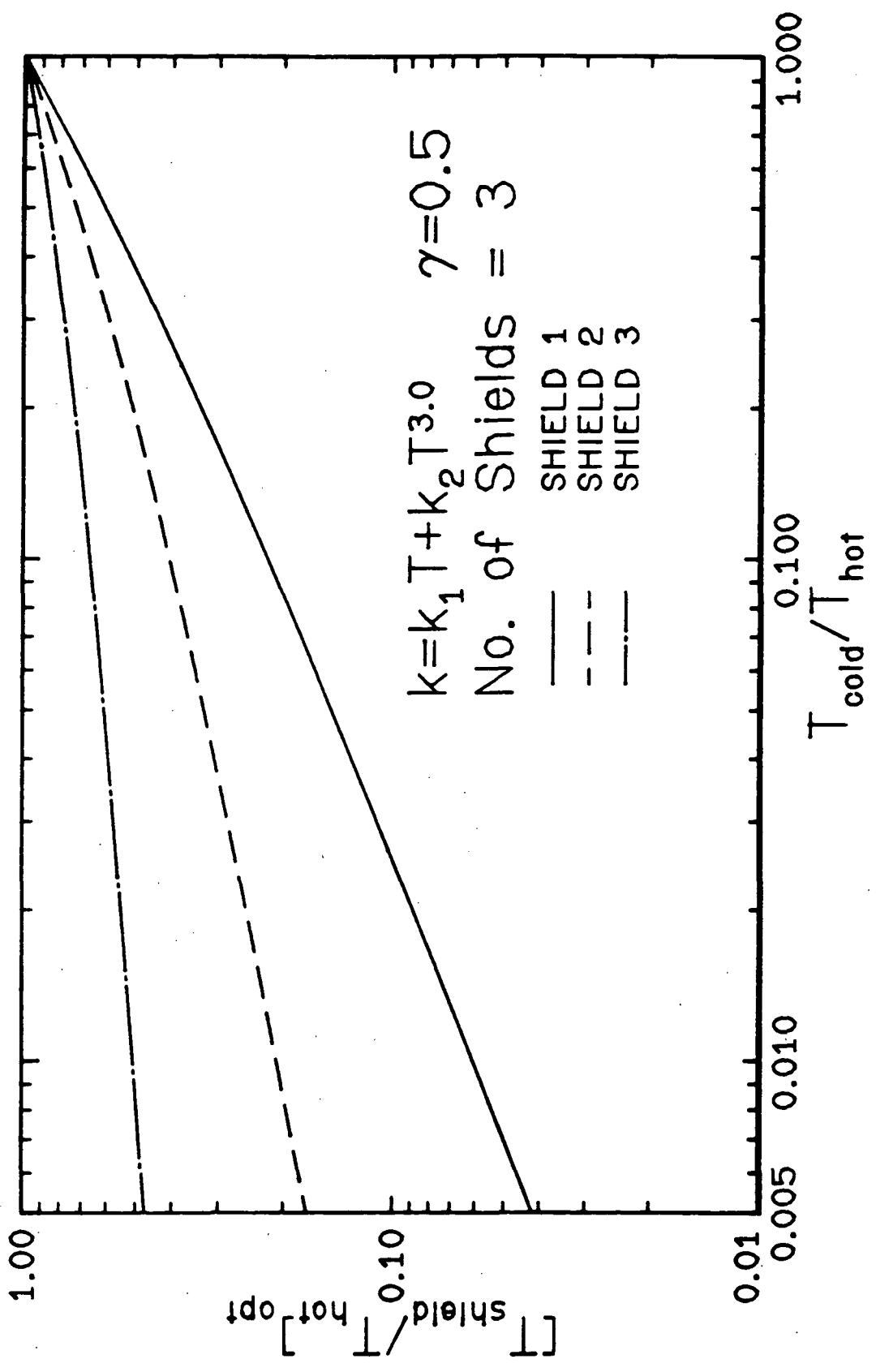


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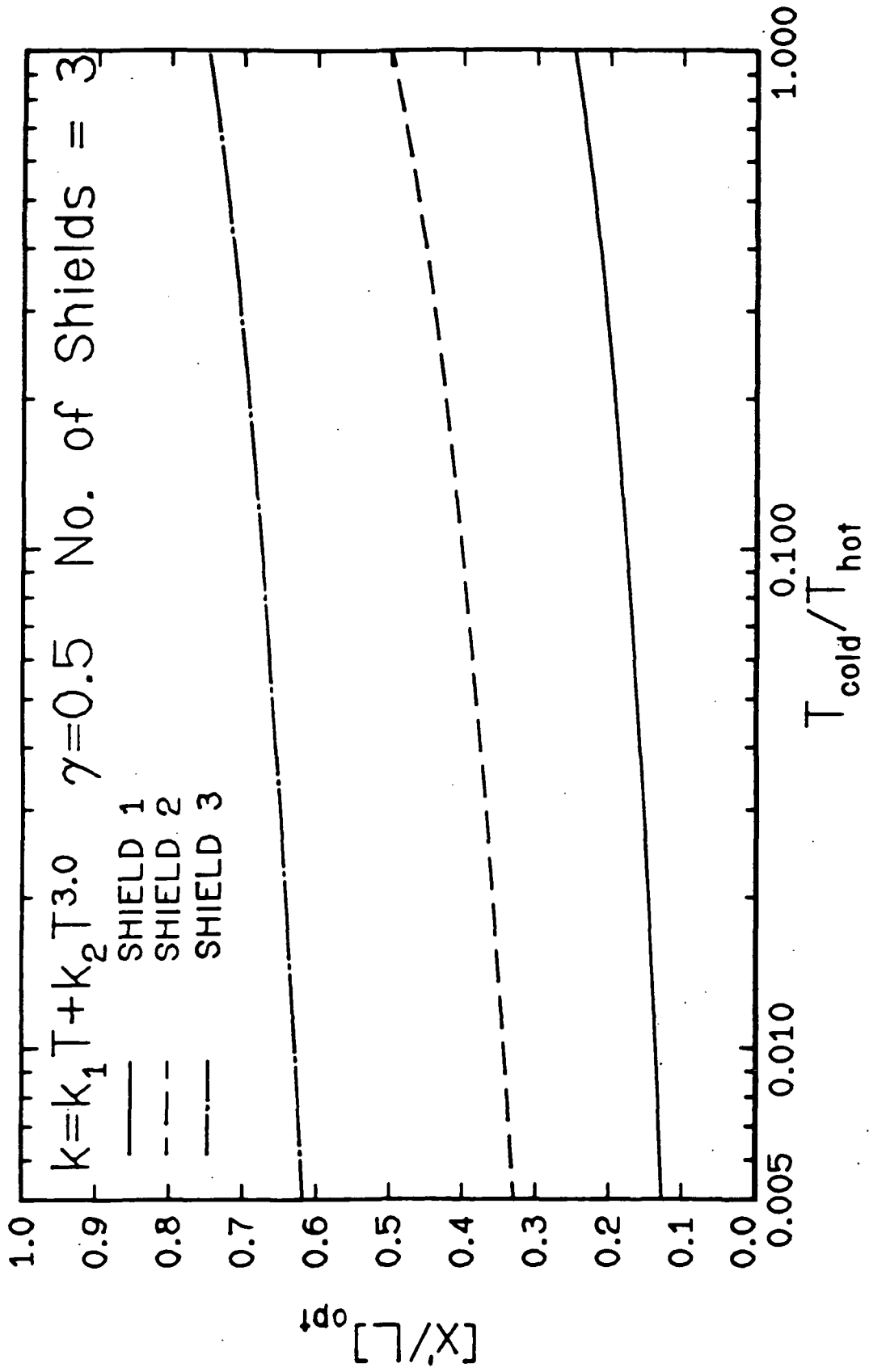


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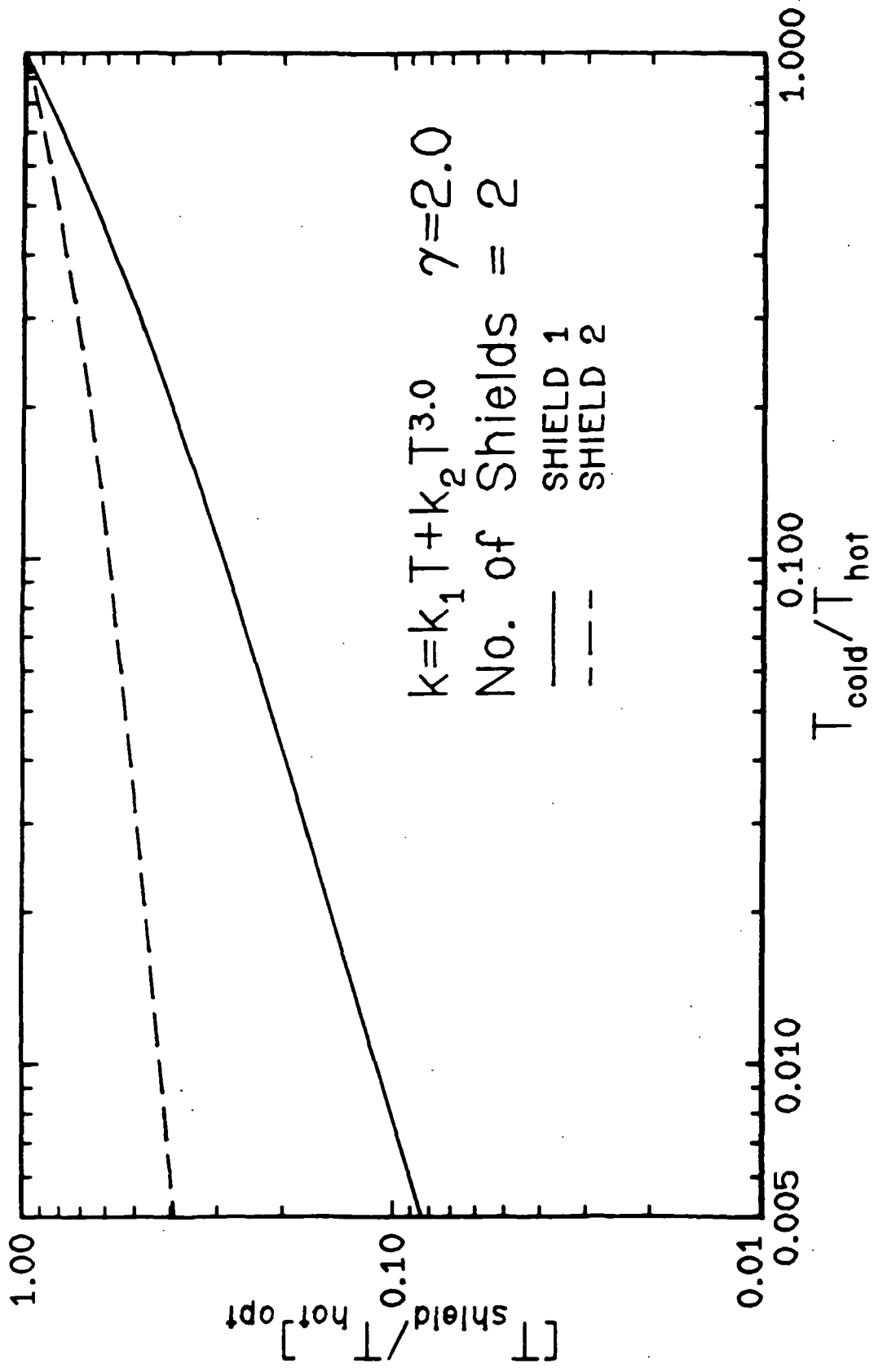


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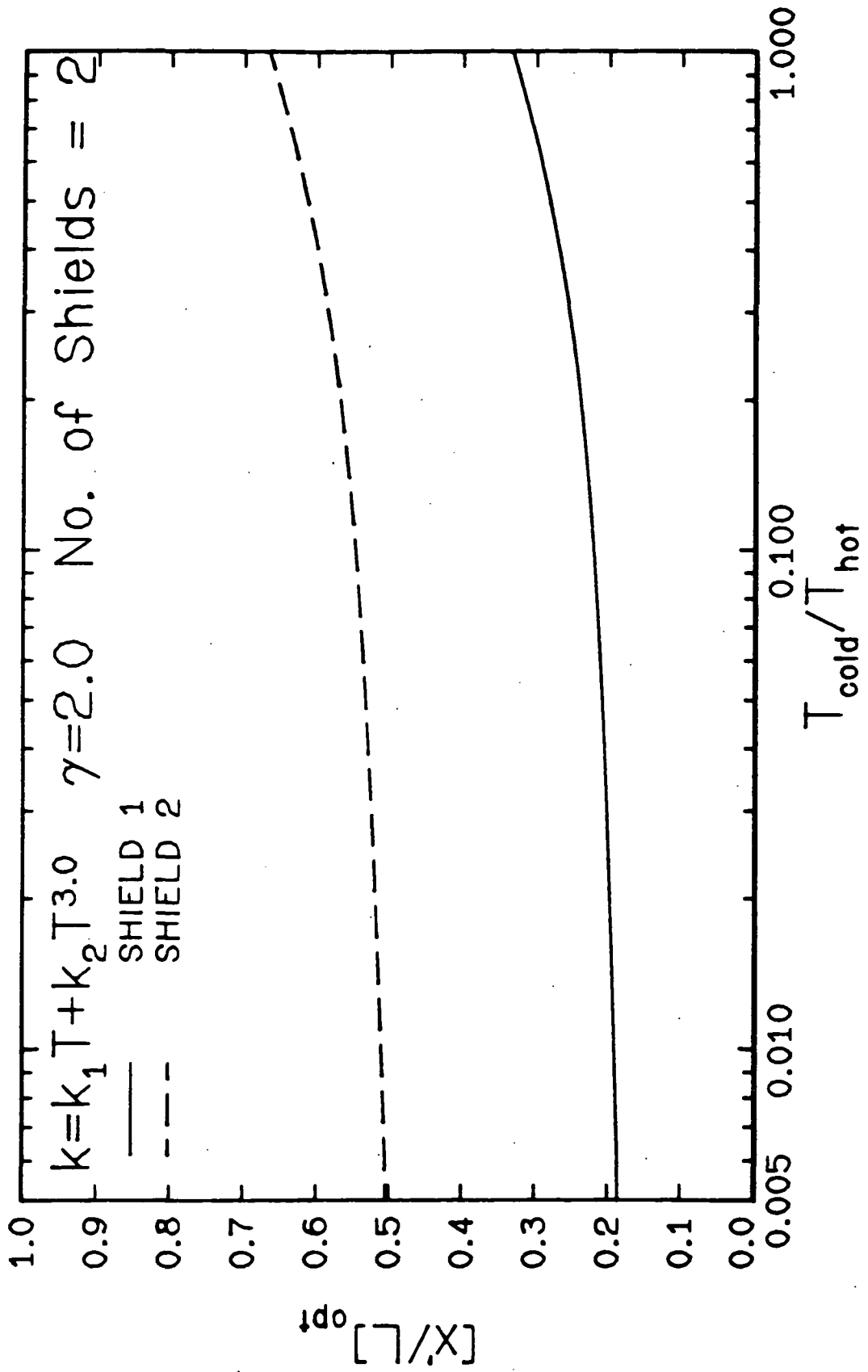


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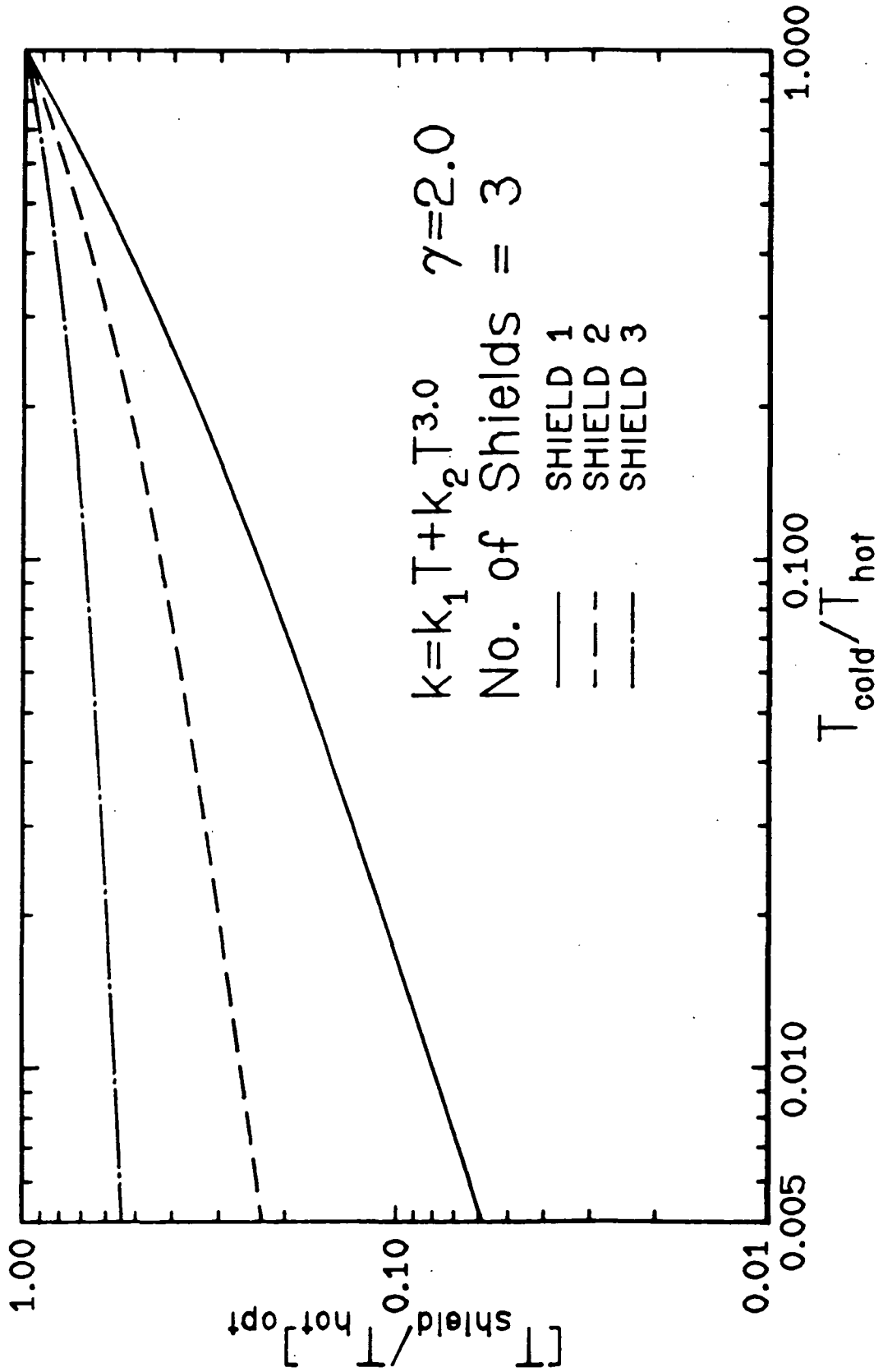


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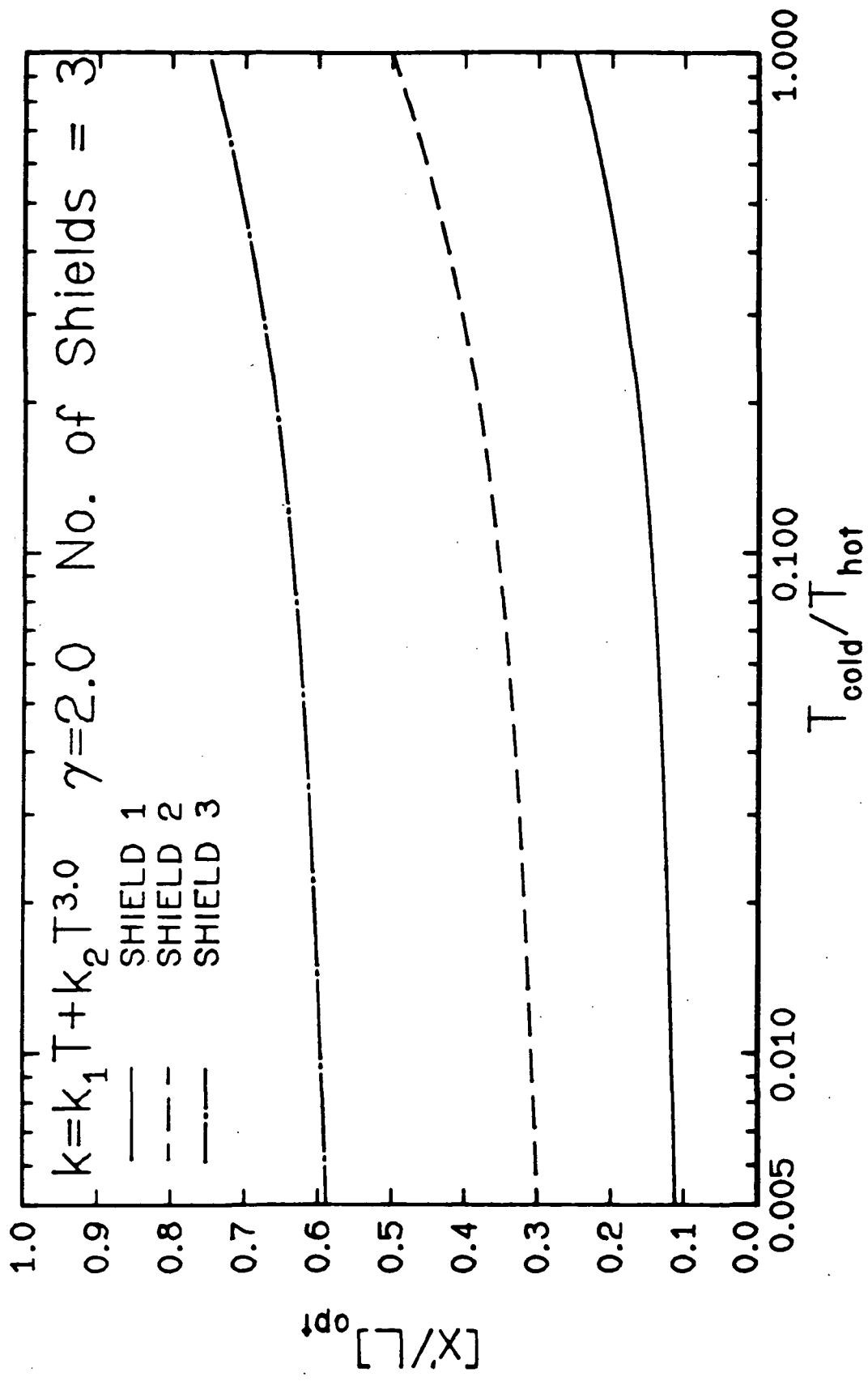


Figure 31

Curve Set 3: Figures 32 through 35
System sensitivity to deviations from the optimum shield
temperatures and locations for two overall temperature ratios
with one cooled shield

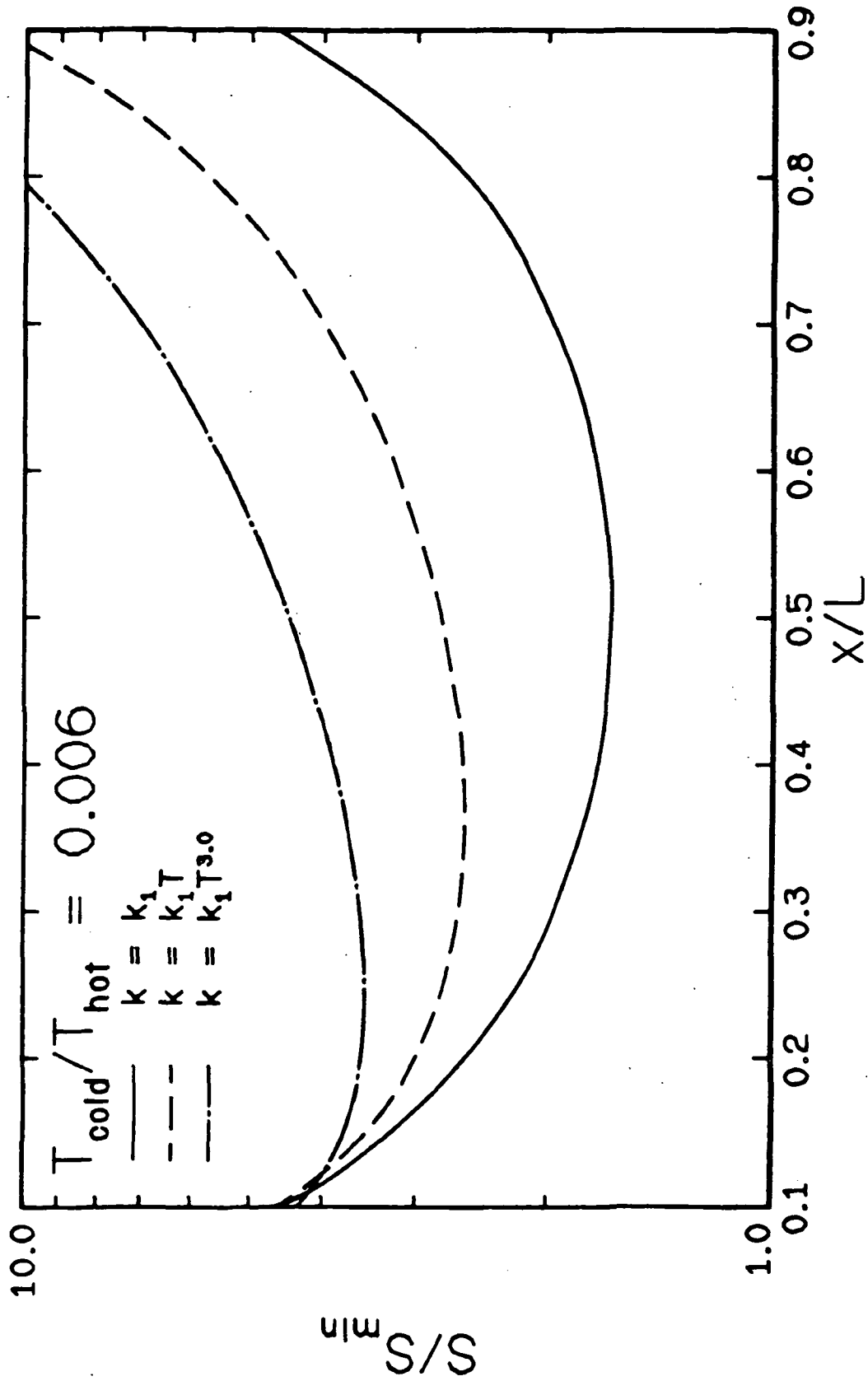


Figure 32

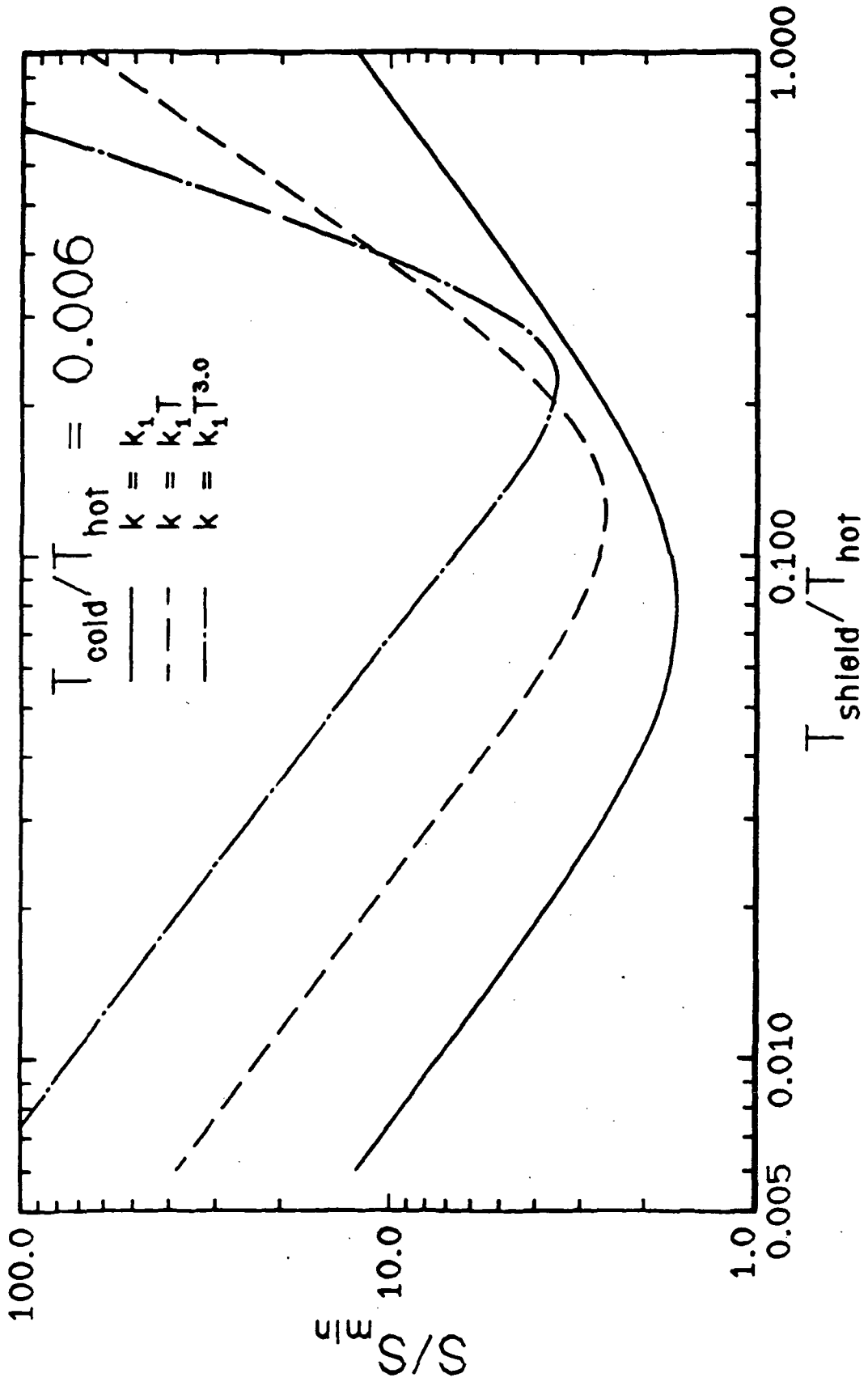


Figure 33

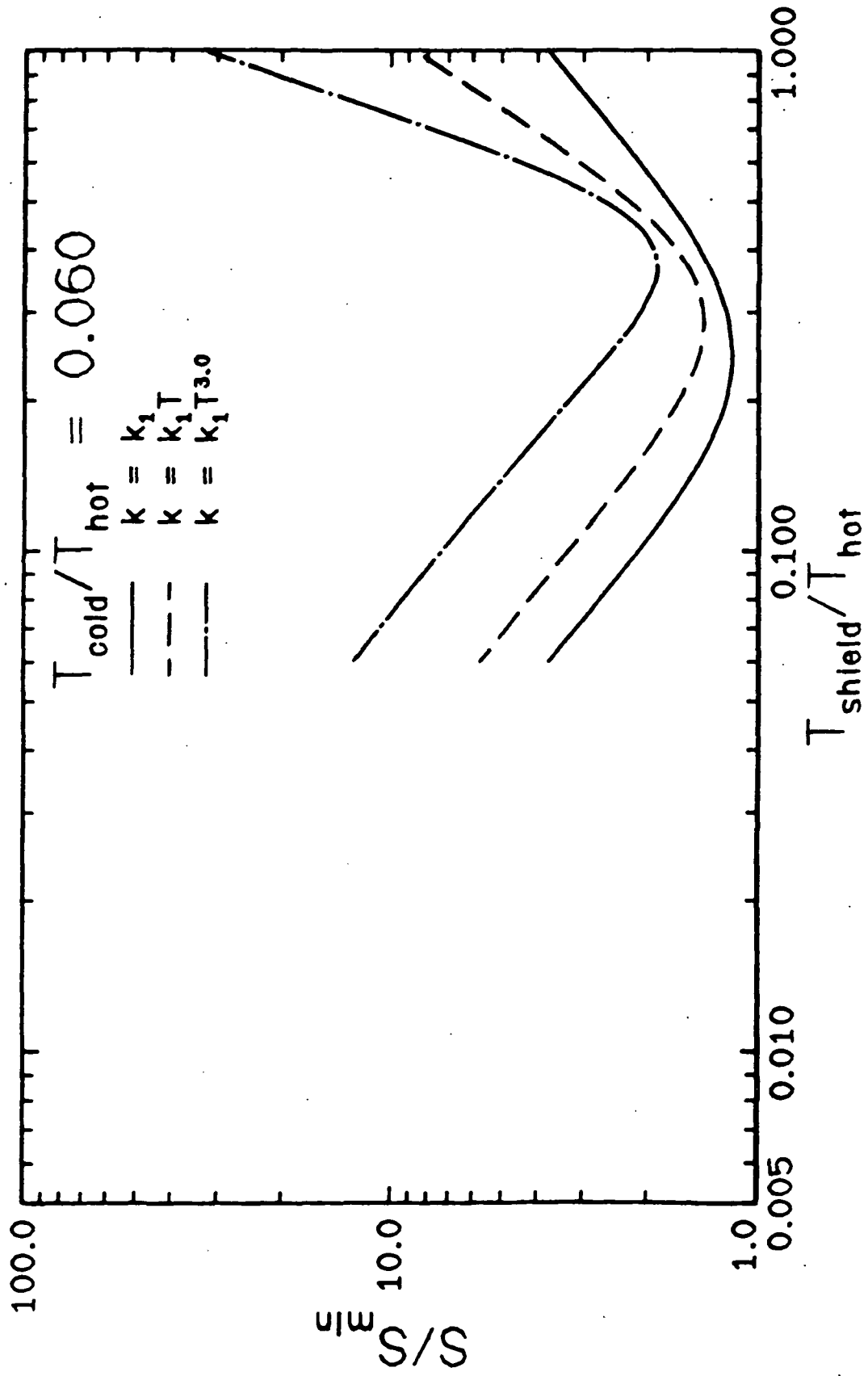


Figure 34

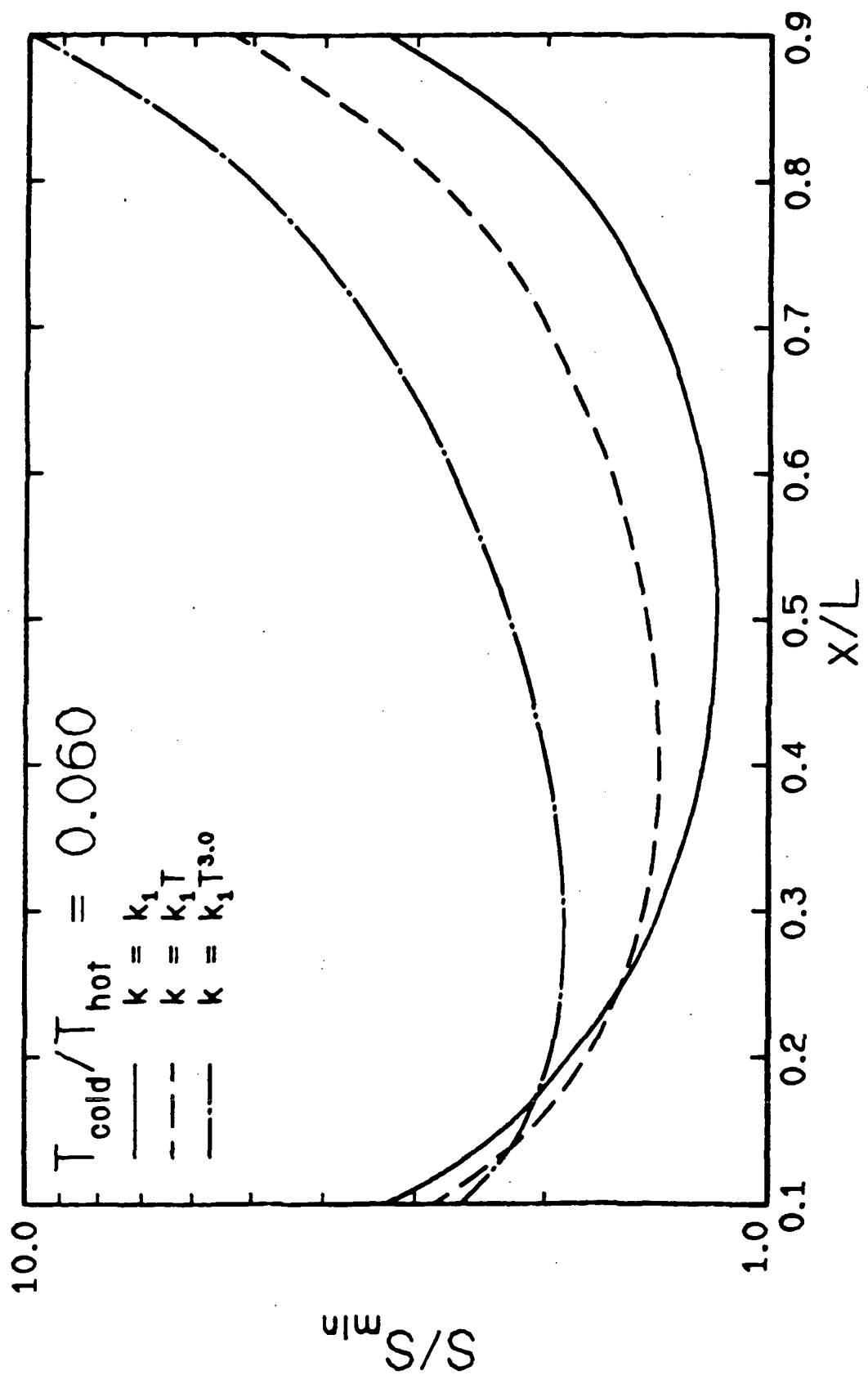


Figure 35

APPENDIX
COMPUTER PROGRAMS

SEPARS and SHIELD

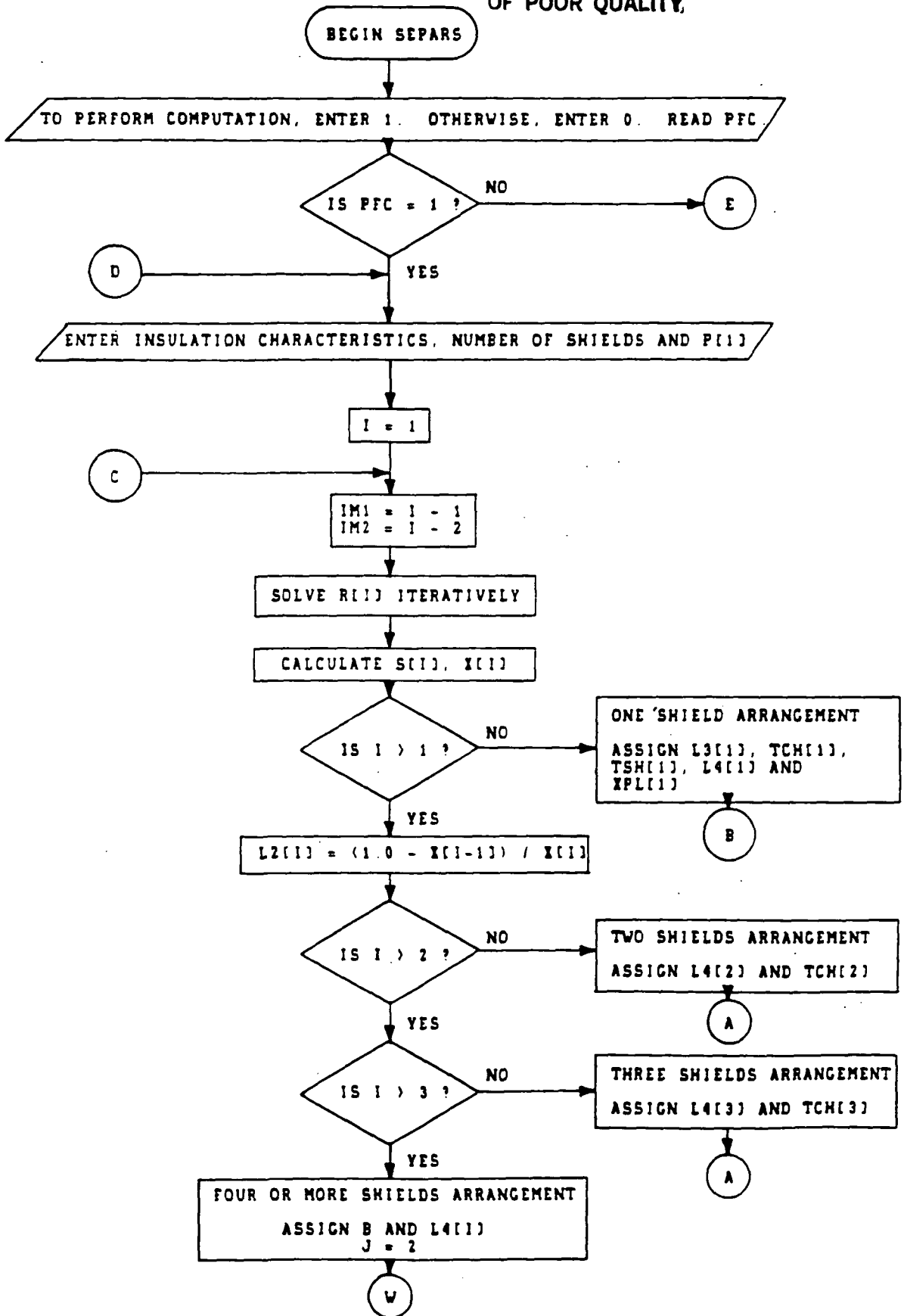
These two programs are essentially identical, but SEPARS is written in PASCAL whereas SHIELD is in BASIC.

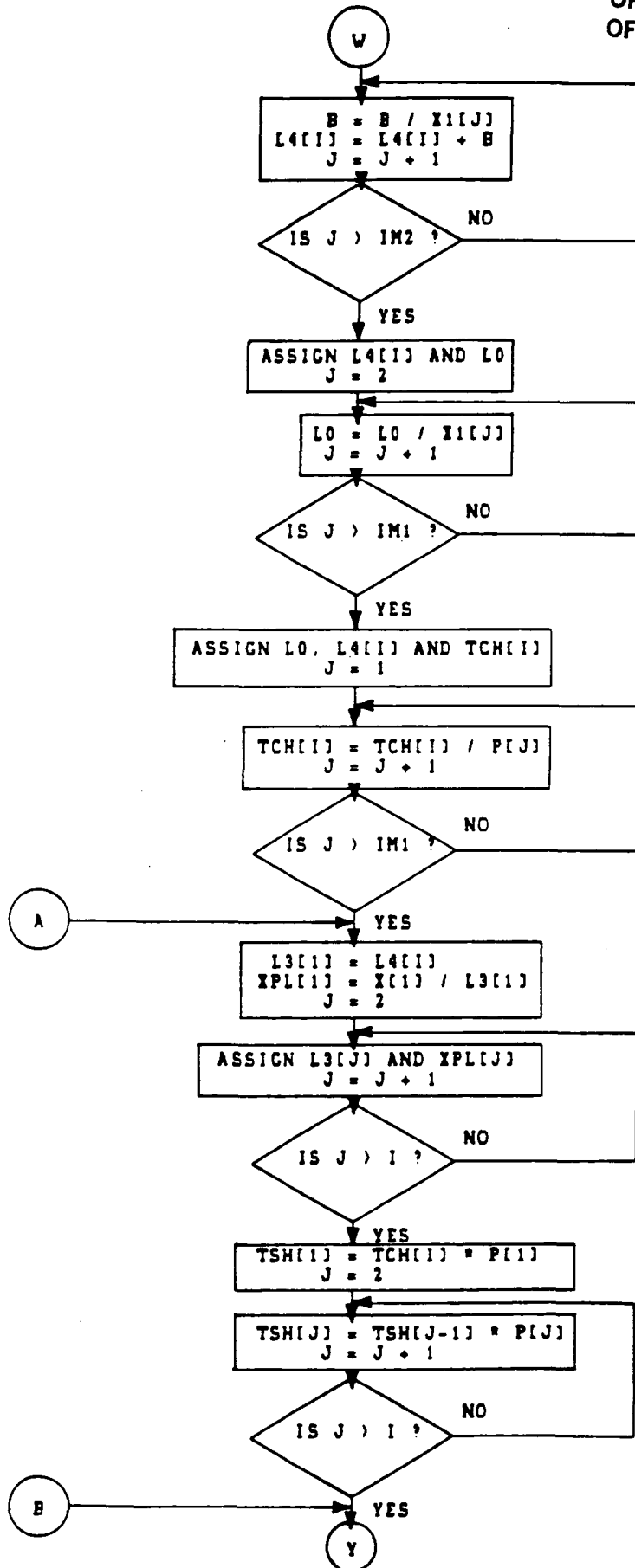
To allow for consecutive calculations of different systems, the program always recycles to the starting point. Consequently, the first input requested is either a 1, if a calculation is to be performed, or a 0, if no more work is to be done.

Next the program requests input of the insulation's characteristics, specifically, the two exponents of the temperatures in the two-term conductivity function, the maximum number of cooled shields (≤ 10) to evaluate, the value of γ , and the temperature ratio of the first shield to the cold wall, $P(1) = T_{S1}/T_C$. The program calculates and presents the characteristics of all optimal systems of cooled shields from one shield to the maximum number specified in the input.

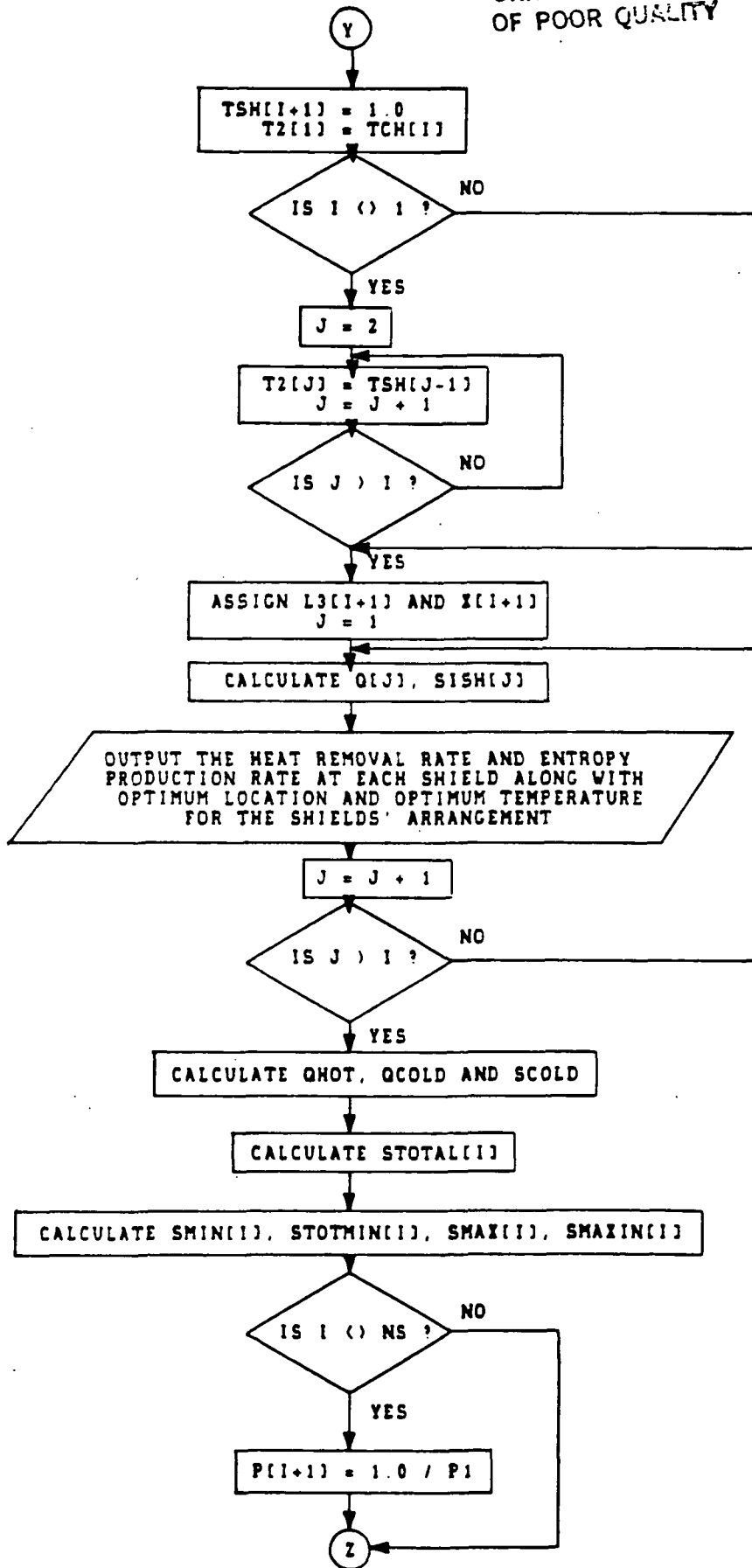
The flow chart and a program sample follows.

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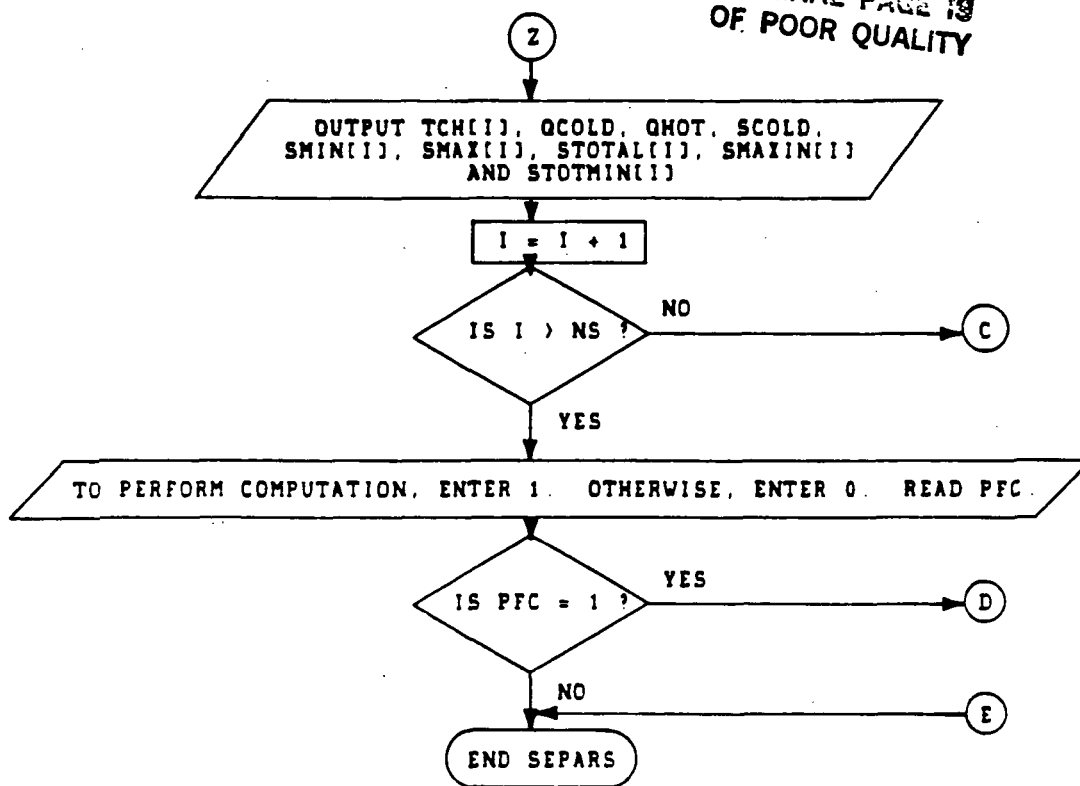


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```

75      B           : REAL.      (* DUMMY VARIABLE *)
76      CC          : REAL.      (* DUMMY VARIABLE *)
77      COUNT       : INTEGER;    (* NUMBER OF ITERATIONS NEEDED TO DETERMINE R(I) *)
78      DD          : REAL;      (* DUMMY VARIABLE *)
79      G, G1       : REAL;      (* DUMMY VARIABLES *)
80      GAMA        : REAL.      (* (K1*(M+1))/(K1*(M+1)) * THOT**(N-M) , ALWAYS > 0 *)
81                                     (* WHERE THOT IS THE HOT WALL TEMPERATURE (K) *)
82      I, IM1, IM2, J : INTEGER,  (* INDICES FOR LOOPS *)
83      JMK         : TEXT.      (* OUTPUT FILE TO BE USED IF DESIRED *)
84      LG          : REAL.      (* DUMMY VARIABLE *)
85      M           : REAL.      (* 1ST POWER IN THE THERMAL CONDUCTIVITY EQUATION *)
86      MP1        : REAL.      (* EQUALS M-1 *)
87      N           : REAL.      (* 2ND POWER IN THE THERMAL CONDUCTIVITY EQUATION *)
88      NP1        : REAL.      (* EQUALS N-1 *)
89      NS         : INTEGER.    (* NUMBER OF SHIELDS *)
90      PFC        : INTEGER.    (* PROGRAM FLOW CONTROLLER *)
91      P1         : REAL.      (* I-TH SHIELD / LOCAL HOT TEMPERATURE RATIO, ALWAYS > 1 *)
92      QCOLD      : REAL.      (* HEAT OUT AT COLD WALL *)
93      QHOT       : REAL.      (* HEAT IN AT HOT WALL *)
94      SCOLD      : REAL.      (* ENTROPY PRODUCTION RATE AT COLD WALL *)
95      U, V       : REAL.      (* DUMMY VARIABLES *)
96      W1, W2, W3 : REAL.      (* DUMMY VARIABLES *)
97      Z1, Z2     : REAL.      (* DUMMY VARIABLES *)
98
99
100
101

```

PROCEDURE INPUT.

```

102 BEGIN
103     (* INPUT OF DATA HEADING *)
104     WRITELN
105     WRITELN( 'ENTER ----) M N NS GAMA P1: (----)',
106     WRITELN( ' ),
107     WRITELN( 'WHERE M ---- 1ST POWER IN THE THERMAL CONDUCTIVITY EQUATION',
108     WRITELN( 'N ---- 2ND POWER IN THE THERMAL CONDUCTIVITY EQUATION',
109     WRITELN( 'NS ---- NUMBER OF SHIELDS',
110     WRITELN( 'GAMA -- =0 IF USING ONE TERM THERMAL CONDUCTIVITY EQUATION',
111     WRITELN( '          >0 IF USING TWO TERM THERMAL CONDUCTIVITY EQUATION',
112     WRITELN( 'P1: -- 1ST SHIELD / COLD WALL TEMPERATURE RATIO, ALWAYS > 1',
113     WRITELN( ' )
114 END
115
116
117

```

PROCEDURE PFCH.

```

118 BEGIN
119     (* PFCH *)
120     WRITELN
121     WRITELN( 'TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.',
122     WRITELN
123 END
124
125
126

```

PROCEDURE SINGLESPEAC.

```

127 BEGIN
128     (* SINGLE SPACE IN OUTPUT *)
129     WRITELN( ' )
130 END
131
132
133

```

FUNCTION PWR(XI,E REAL) REAL.

```

134 VAR
135     A
136     REAL
137 BEGIN
138     (* COMPUTE XI**E *)
139     A =E*LN(XI),
140     PWR =EXP(A)
141 END
142
143

```

FUNCTION D(E, XI, REAL) REAL.

```

144 BEGIN
145     (* FUNCTIONAL D *)
146     D =(E-1.0)*PWR(XI,E)-E/(PWR(XI,(1.0-E)))-(1.0/SQR(XI))
147 END
148
149
150

```

```

151 FUNCTION F(E,XX:REAL):REAL;
152 BEGIN
153   F=(PWR(XX,(E+1.0))-PWR(XX,E)-1.0*(1.0/XX)
154   END;
155
156
157
158 FUNCTION SIMPSON(TCHR:REAL):REAL;
159 TYPE
160   ARR=ARRAY[1:101] OF REAL;
161
162 VAR
163   C,Y           ARR;
164   DELTAT        REAL;
165   H             REAL;
166   K,L          INTEGER;
167
168 BEGIN
169   (* COMPUTE MINIMUM ENTROPY PRODUCTION RATE USING SIMPSON'S NUMERICAL INTEGRATION SCHEME *)
170   DELTAT=(1.0-TCHR)/100.0;
171   FOR L=1 TO 101 DO
172     BEGIN
173       C(L)=TCHR+DELTAT*(L-1);
174       Y(L)=PWR((PWR(C(L),N)+GAMA*NP1/MP1)*PWR(C(L),N),0.5)/C(L);
175     END;
176   H=Y(1)+Y(101);
177   FOR K=2 TO 100 DO
178     BEGIN
179       IF K=((K DIV 2)*2) THEN
180         H=H+4.0*Y(K);
181       ELSE
182         H=H+2.0*Y(K);
183     END;
184   SIMPSON=(SOR(DELTAT/3.0*H))/((1.0-GAMA*NP1/MP1);
185   END;
186
187
188
189
190
191
192   (* MAIN PROGRAM BODY *)
193
194 BEGIN
195   PEGN;
196   READLN;
197   READ(PFC);
198   WHILE PFC=1 DO
199     BEGIN
200
201       (* THIS BLOCK IS USED TO INPUT THE INSULATION THERMAL CONDUCTIVITY, NUMBER *)
202       (* OF SHIELDS AND 1ST. SHIELD / COLD WALL TEMPERATURE RATIO *)
203
204       INPUT H;
205       READLN;
206       READ(M,N,MS,GAMA,P(1));
207       SINGLESPEACE;
208       IF GAMA=0.0 THEN
209         WRITELN(' THERMAL CONDUCTIVITY OF THE INSULATION IS K = K1*T**',M,3,1)
210       ELSE
211         BEGIN
212           WRITELN(' THERMAL CONDUCTIVITY OF THE INSULATION IS K = K1*T**',M,3,1,' + K2*T**',M,3,1),
213           WRITELN(' (K2*(M+1))/(K1*(M+1))*THOT**'(M-M) = ',GAMA,9,2)
214         END;
215       SINGLESPEACE;
216       SINGLESPEACE;
217
218       NP1=M+1.0;
219       MP1=N+1.0;
220       FOR I=1 TO MS DO
221         BEGIN
222           IM1=I-1;
223           IM2=I-2;
224           R(I)=0.000001;
225           CC=0.1;
226           DD=1.0;
227           COUNT=0;

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230      (* THIS BLOCK CALCULATES R(I) ITERATIVELY *)
231
232      REPEAT
233      P1 = P(I)*R(I),
234      W1 = PWR(R(I),M)*F(M,P(I))+GAMA*PWR(R(I),M)*F(M,P(I)),
235      W2 = SQR(PWR(R(I),(M-1.0))*D(M,P(I))+GAMA*PWR(R(I),(M-1.0))*D(M,P(I))),
236      W3 = -SQR(D(M,P1)+GAMA*D(M,P1))/(F(M,P1)+GAMA*F(M,P1)),
237      G = (W2/W1)+W3,
238      G1 = C*DD,
239      IF G1<0.0 THEN GOTO 100,
240      IF G1=0.0 THEN GOTO 200,
241      CC = (-0.1)*CC,
242      IF ABS(CC)<0.000001 THEN GOTO 200,
243      DD = -DD,
244      100 R(I) = R(I)+CC,
245      IF (R(I)>0.999999) OR (R(I)<0.000001) THEN
246      BEGIN
247      R(I) = R(I)-0.9*CC,
248      CC = 0.1*CC
249      END,
250      200 COUNT = COUNT+1,
251      UNTIL (G1=0.0) OR (ABS(CC)<0.000001);
252
253
254      U = (PWR(R(I),(M-1.0))*D(M,P(I))+GAMA*PWR(R(I),(M-1.0))*D(M,P(I))),
255      X(I) = U/(D(M,P1)+GAMA*D(M,P1)),
256      X(1) = X(I)/(1.0-X(I)),
257      V = (F(M,P1)+GAMA*F(M,P1))/(1.0-X(I)),
258      S(I) = V*(PWR(R(I),M)*F(M,P(I))+GAMA*PWR(R(I),M)*F(M,P(I)))/X(I),
259      S(1) = S(I)/(1.0+GAMA*MP1/MP1)/MP1,
260
261      (* IN THIS BLOCK VARIABLES ARE ASSIGNED FOR DIFFERENT SHIELD CONFIGURATIONS *)
262
263      IF I=1 THEN
264      BEGIN
265      L3(I) = (1.0-X(I))/X(I),
266      IF I=2 THEN
267      IF I=3 THEN
268      BEGIN
269      B = 1.0,
270      L4(I) = 0.0,
271      FOR J = 2 TO IM2 DO
272      BEGIN
273      B = B/X(I(J)),
274      L4(I) = L4(I)+B
275      END,
276      L4(I) = L4(I)*(1.0-X(I))+1.0,
277      L0 = 1.0-X(I),
278      FOR J = 2 TO IM1 DO L0 = L0/X(I(J)),
279      L0 = L0/X(I),
280      L4(I) = L4(I)+L0,
281      TCH(I) = R(I),
282      FOR J = 1 TO IM1 DO TCH(I) = TCH(I)/P(J),
283      END
284      ELSE
285      BEGIN
286      L4(I) = 1.0+(1.0-X(I))*(1.0-X(I))/(X(I)*X(I)),
287      TCH(I) = R(I)/(P(I)*P(I))
288      END
289      ELSE
290      BEGIN
291      L4(I) = X(I)+(1.0-X(I))/X(I),
292      TCH(I) = R(I)/P(I)
293      END,
294      L3(I) = L4(I),
295      XPL(I) = X(I)/L3(I),
296      FOR J = 2 TO I DO
297      BEGIN
298      L3(J) = L3(J-1)/L3(J),
299      XPL(J) = XPL(J-1)*X(J)/L3(J)
300      END,
301      TSH(I) = TCH(I)*P(I),
302      FOR J = 2 TO I DO TSH(J) = TSH(J-1)*P(J)
303      END
304      ELSE

```

```

305      BEGIN
306      L3(I) = 1.0.
307      TCH(I) = R(I).
308      TSH(I) = TCH(I)*P(I).
309      L4(I) = 1.0.
310      XPL(I) = X(I)
311      END.
312      TSH(I+1) = 1.0.
313      T2(I) = TCH(I).
314
315
316      SINGLESPACE.
317      WRITELN('      NUMBER OF SHIELDS = ', I-2).
318      WRITELN('      NUMBER OF ITERATIONS = ', COUNT).
319      SINGLESPACE.
320      SINGLESPACE.
321      WRITELN('      HEAT REMOVAL      ENTROPY PRODUCTION      OPTIMUM      OPTIMUM')
322      WRITELN('      RATE              RATE              LOCATION      TEMPERATURE')
323      WRITELN('      -----      -----      -----      -----')
324      SINGLESPACE.
325      IF I<=1 THEN
326      FOR J = 2 TO I DO T2(J) = TSH(J-1).
327      L3(I+1) = L3(I).
328      X(I+1) = 0 - X(I).
329
330      (* IN THIS BLOCK DIMENSIONLESS HEAT REMOVAL AND ENTROPY PRODUCTION RATES *)
331      (* ARE CALCULATED FOR EACH SHIELD *)
332
333      FOR J = 1 TO I DO
334      BEGIN
335      Z1 = ((PWR(TSH(I+1), NP1) - PWR(TSH(I), NP1)) * L3(J+1) / X(J+1) - (PWR(TSH(J), NP1) - PWR(T2(J), NP1)) * L3(J) / X(J)) / NP1.
336      Z2 = ((PWR(TSH(J+1), NP1) - PWR(TSH(J), NP1)) * L3(J+1) / X(J+1) - (PWR(TSH(J), NP1) - PWR(T2(J), NP1)) * L3(J) / X(J)) / NP1.
337      Q(I) = (Z1 * GAMA * Z2) / (1.0 + GAMA * NP1 / MP1).
338      SISH(J) = Q(I) / TSH(J).
339      WRITELN('      SHIELD ', J, ' : ', S, Q(I), ' : ', S, SISH(J), ' : ', S, XPL(J), ' : ', S, TSH(J), ' : ', S).
340      END.
341
342      (* FINALLY, OTHER QUANTITIES OF INTEREST ARE CALCULATED IN THIS BLOCK *)
343
344      SINGLESPACE.
345      QHOT = ((1.0 - PWR(TSH(I), NP1)) * GAMA - GAMA * PWR(TSH(I), NP1)) * L3(I) / (X(I+1) * MP1)) / (1.0 + GAMA * NP1 / MP1).
346      QCOLD = (PWR(TSH(I), NP1) - PWR(TCH(I), NP1)) * GAMA * PWR(TSH(I), NP1) - GAMA * PWR(TCH(I), NP1)) * L3(I) / (X(I) * MP1).
347      QCOLD = QCOLD / (1.0 + GAMA * NP1 / MP1).
348      SCOLD = QCOLD / TCH(I).
349      STOTAL(I) = SCOLD - QHOT.
350      FOR J = 1 TO I DO STOTAL(I) = STOTAL(I) + SISH(J).
351      SHMIN(I) = SIMPSON(TCH(I)).
352      STOTMIN(I) = STOTAL(I) / SHMIN(I).
353      SHMAX(I) = ((1.0 - PWR(TCH(I), NP1)) * GAMA - GAMA * PWR(TCH(I), NP1)) * (1.0 / TCH(I) - 1.0) / MP1) / (1.0 + GAMA * NP1 / MP1).
354      SHAIN(I) = SHMAX(I) / SHMIN(I).
355
356
357      IF I<=1 THEN P(I+1) = 1.0 / P(I).
358      SINGLESPACE.
359      WRITELN('      COLD WALL / HOT WALL TEMPERATURE RATIO = ', TCH(I), ' : ', S, 14.6).
360      WRITELN('      HEAT OUT AT COLD WALL = ', QCOLD, ' : ', S, 14.6).
361      WRITELN('      HEAT IN AT HOT WALL = ', QHOT, ' : ', S, 14.6).
362      WRITELN('      ENTROPY PRODUCTION RATE AT COLD WALL = ', SCOLD, ' : ', S, 14.6).
363      WRITELN('      ENTROPY PRODUCTION RATE AT HOT WALL = ', -QHOT, ' : ', S, 14.6).
364      WRITELN('      MINIMUM ENTROPY PRODUCTION RATE = ', SHMIN(I), ' : ', S, 14.6).
365      WRITELN('      MAXIMUM ENTROPY PRODUCTION RATE = ', SHMAX(I), ' : ', S, 14.6).
366      WRITELN('      TOTAL ENTROPY PROD RATE WITH ', I-2, ' SHIELDS = ', STOTAL(I), ' : ', S, 14.6).
367      WRITELN('      MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO = ', SHAIN(I), ' : ', S, 14.6).
368      WRITELN('      TOTAL / MINIMUM ENTROPY PRODUCTION RATIO = ', STOTMIN(I), ' : ', S, 14.6).
369      SINGLESPACE.
370      SINGLESPACE.
371      SINGLESPACE.
372
373      END.
374      PFCB.
375      READLN.
376      READ(PFC)
377      END
378      END
379      /EOP

```

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 1

ENTER ----> M N NS GAMA PC1] <----

WHERE: M ----- 1ST. POWER IN THE THERMAL CONDUCTIVITY EQUATION
 N ----- 2ND. POWER IN THE THERMAL CONDUCTIVITY EQUATION
 NS ---- NUMBER OF SHIELDS
 GAMA -- =0 IF USING ONE TERM THERMAL CONDUCTIVITY EQUATION
 >0 IF USING TWO TERM THERMAL CONDUCTIVITY EQUATION
 PC1] -- 1ST. SHIELD / COLD WALL TEMPERATURE RATIO, ALWAYS > 1

? 1.0 3.0 1 2.5 15.0

THERMAL CONDUCTIVITY OF THE INSULATION IS $K = K1*T^{1.0} + K2*T^{3.0}$
 $[K2*(M+1)]/[K1*(N+1)]*THOT*(N-M) = 2.50$

NUMBER OF SHIELDS = 1
 NUMBER OF ITERATIONS = 35

	HEAT REMOVAL RATE	ENTROPY PRODUCTION RATE	OPTIMUM LOCATION	OPTIMUM TEMPERATURE
	-----	-----	-----	-----
SHIELD 1	0.43837	1.85659	0.36744	0.23611

COLD WALL / HOT WALL TEMPERATURE RATIO = 0.015741
 HEAT OUT AT COLD WALL = 0.014350
 HEAT IN AT HOT WALL = 0.452719
 ENTROPY PRODUCTION RATE AT COLD WALL = 0.911631
 ENTROPY PRODUCTION RATE AT HOT WALL = -0.452719
 MINIMUM ENTROPY PRODUCTION RATE = 1.000503
 MAXIMUM ENTROPY PRODUCTION RATE = 18.236148
 TOTAL ENTROPY PROD. RATE WITH 1 SHIELDS = 2.315503
 MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO = 18.226922
 TOTAL / MINIMUM ENTROPY PRODUCTION RATIO = 2.314340

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 1

ENTER ----> M N NS GAMA PC1] <----

WHERE: M ----- 1ST. POWER IN THE THERMAL CONDUCTIVITY EQUATION
 N ----- 2ND. POWER IN THE THERMAL CONDUCTIVITY EQUATION
 NS ---- NUMBER OF SHIELDS
 GAMA -- =0 IF USING ONE TERM THERMAL CONDUCTIVITY EQUATION
 >0 IF USING TWO TERM THERMAL CONDUCTIVITY EQUATION
 PC1] -- 1ST. SHIELD / COLD WALL TEMPERATURE RATIO, ALWAYS > 1

? 1.0 .090 2 0.0 25.0

THERMAL CONDUCTIVITY OF THE INSULATION IS $K = K1*T^{1.0}$

NUMBER OF SHIELDS = 1
 NUMBER OF ITERATIONS = 23

	HEAT REMOVAL RATE	ENTROPY PRODUCTION RATE	OPTIMUM LOCATION	OPTIMUM TEMPERATURE
	-----	-----	-----	-----
SHIELD 1	0.75466	7.03151	0.35870	0.10732

COLD WALL / HOT WALL TEMPERATURE RATIO	=	0.004293
HEAT OUT AT COLD WALL	=	0.016030
HEAT IN AT HOT WALL	=	0.770687
ENTROPY PRODUCTION RATE AT COLD WALL	=	3.734070
ENTROPY PRODUCTION RATE AT HOT WALL	=	-0.770687
MINIMUM ENTROPY PRODUCTION RATE	=	3.504633
MAXIMUM ENTROPY PRODUCTION RATE	=	115.966533
TOTAL ENTROPY PROD. RATE WITH 1 SHIELDS	=	9.994893
MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO	=	33.089491
TOTAL / MINIMUM ENTROPY PRODUCTION RATIO	=	2.851908

NUMBER OF SHIELDS = 2
NUMBER OF ITERATIONS = 36

	HEAT REMOVAL RATE	ENTROPY PRODUCTION RATE	OPTIMUM LOCATION	OPTIMUM TEMPERATURE
	-----	-----	-----	-----
SHIELD 1	0.05470	2.71297	0.17465	0.02016
SHIELD 2	0.88421	4.70678	0.48690	0.18786

COLD WALL / HOT WALL TEMPERATURE RATIO	=	0.000806
HEAT OUT AT COLD WALL	=	0.001162
HEAT IN AT HOT WALL	=	0.940073
ENTROPY PRODUCTION RATE AT COLD WALL	=	1.440716
ENTROPY PRODUCTION RATE AT HOT WALL	=	-0.940073
MINIMUM ENTROPY PRODUCTION RATE	=	3.921467
MAXIMUM ENTROPY PRODUCTION RATE	=	619.477774
TOTAL ENTROPY PROD. RATE WITH 2 SHIELDS	=	7.920388
MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO	=	157.970919
TOTAL / MINIMUM ENTROPY PRODUCTION RATIO	=	2.019751

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 0

0.175 CP SECS, 12415B CM USED.

PROGRAM SHIELD

```

1 00010 REM THIS IS A "BASIC" PROGRAM TO CALCULATE OPTIMUM TEMPERATURES,
2 00020 REM LOCATIONS, AND COOLING LOADS FOR COOLED SHIELDS IN A CRYOGENIC
3 00030 REM INSULATION SYSTEM WHOSE THERMAL CONDUCTIVITY FOLLOWS THE RELATION
4 00040 REM  $k=C_1T^M_0 + C_2T^N_0$ 
5 00045 REM MODIFIED IN LATE NOV. 1982.
6 00050 REM
7 00060 REM DEFINITION OF SYMBOLS USED:
8 00070 REM
9 00080 REM COLD-SIDE WALL TEMPERATURE T0
10 00090 REM WARM-SIDE WALL TEMPERATURE T9
11 00100 REM SPACING BETWEEN SHIELDS AT I+1 AND I-1 L1(I)
12 00110 REM OVERALL THICKNESS OF INSULATION L
13 00120 REM LOCAL SPACING RATIO, L1(I)/L1(I-1), L2(I)
14 00130 REM OVERALL SPACING RATIO, L/L1(1), L4(I)
15 00140 REM (DISTANCE FROM COLD WALL)/L LS(I)
16 00150 REM I-TH SHIELD TEMPERATURE T(I)
17 00160 REM I-TH SHIELD POSITION RATIO X(I)
18 00170 REM I-TH SHIELD TEMPERATURE RATIO P(I) (ALWAYS >1)
19 00180 REM I-TH COLD-WARM TEMPERATURE RATIO R(I) (ALWAYS >1)
20 00190 REM I-TH DIMENSIONLESS ENTROPY PRODUCTION RATE S(I)
21 00195 REM I-TH DIMENSIONLESS HEAT REMOVAL RATE Q(I)
22 00210 REM TOTAL DIMENSIONLESS ENTROPY PROD. RATE S2(I)
23 00220 REM MINIMUM ENTROPY PRODUCTION RATE S0(I)
24 00230 REM ENTROPY PROD. RATE WITHOUT SHIELDS S9(I)
25 00240 REM ENTROPY PROD. RATE RATIOS S3=S2/S0 AND S4=S9/S0
26 00250 REM NUMBER OF SHIELDS M (= OR (10)
27 00260 REM
28 00265 DIM C(10),Y(10)
29 00270 PRINT
30 00280 PRINT "INPUT : IF MORE WORK IS TO BE DONE, 0 IF FINISHED"
31 00290 INPUT A
32 00300 IF A=0 THEN 01350
33 00310 PRINT "INPUT NO.,NO.,M,GAMMA & P(1)"
34 00320 INPUT NO.,NO.,M,CO,P(1)
35 00325 DEF FND(Y)=(NO+1)*Y^NO-NO/(Y^(1-NO))-1/(Y*Y)
36 00335 DEF FNE(Y)=(NO+1)*Y^NO-NO/(Y^(1-NO))-1/(Y*Y)
37 00335 DEF FNF(Y)=Y*(NO+1)-Y^NO-1/Y-1
38 00340 DEF FNG(Y)=Y*(NO+1)-Y^NO-1/Y-1
39 00350 PRINT "EXPONENT NO=","NO." EXPONENT NO=","NO." GAMMA=","CO
40 00355 M1=NO+1
41 00358 M2=NO-1
42 00360 FOR I=1 TO M
43 00370 I1=I-1
44 00380 I2=I+1
45 00390 R(I)=.000001
46 00400 C=1
47 00410 D=1
48 00420 P1=P(1)*R(I)
49 00430 W1=(R(I1)^M1)*FNE(P1)+CO*R(I1)^M2)*FNC(P1)
50 00435 W2=(R(I1)^M1)*FND(P1)+CO*R(I1)^M2)*FNE(P1)^2)
51 00438 W3=-((FND(P1)+CO)*FNE(P1))^2)/(FNE(P1)+CO)*FNC(P1)
52 00439 C=W2/W1+W3
53 00440 C1=C*D
54 00450 IF C1<0 THEN 00500
55 00460 IF C1=0 THEN 00570
56 00470 C=-1*C
57 00480 IF ABS(C) > 0.00001 THEN 00570
58 00490 D=D
59 00500 R(I)=R(I)+C
60 00510 IF R(I) > 0.999999 THEN 00540
61 00520 IF R(I) < 0.000001 THEN 00540
62 00530 GOTO 00420
63 00540 R(I)=R(I)-1*C
64 00550 C=1*C
65 00560 GOTO 00420
66 00570 U=(R(I1)^M1)*FND(P1)+CO*R(I1)^M2)*FNE(P1)
67 00575 X1(I)=U/(FND(P1)+CO)*FNE(P1)
68 00580 X(I)=X1(I)/(1+X1(I))
69 00590 V=(FNE(P1)+CO)*FNC(P1)/(1-X1(I))
70 00595 S(I)=V+(R(I1)^M1)*FNE(P1)+CO*R(I1)^M2)*FNC(P1)/X(I)
71 00596 S(I)=S(I)/(1+CO*M1/M1)/M1
72 00600 IF I=1 THEN 00470
73 00610 L3(I)=1
74 00620 R0(I)=R(I)

```

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```

75 00630 T1(I)=R0(I)*P(I)
76 00640 L4(I)=1
77 00650 L5(I)=X(I)
78 00660 GOTO 00930
79 00670 L2(I)=(1-X(I-1))/X(I)
80 00680 IF I>2 THEN 00720
81 00690 L4(2)=X(1)+(1-X(1))/X(2)
82 00700 R0(2)=R(2)/P(1)
83 00710 GOTO 00930
84 00720 IF I>3 THEN 00740
85 00730 L4(3)=1+(1-X(1))*(1-X(2))/(X(2)*X(3))
86 00740 R0(3)=R(3)/(P(1)*P(2))
87 00750 GOTO 00930
88 00760 E=1
89 00770 L4(I)=0
90 00780 FOR J=2 TO I2
91 00790 B=B/X(I,J)
92 00800 L4(I)=L4(I)+B
93 00810 NEXT J
94 00820 L4(I)=L4(I)*(1-X(I))+1
95 00830 L0=1-X(I)
96 00840 FOR J=2 TO I1
97 00850 L0=L0/X(I,J)
98 00860 NEXT J
99 00870 L0=L0*X(I)
100 00880 L4(I)=L4(I)+L0
101 00890 R0(I)=R(I)
102 00900 FOR J=1 TO I1
103 00910 R0(I)=R0(I)/P(J)
104 00920 NEXT J
105 00930 L3(I)=L4(I)
106 00940 L5(I)=X(I)/L3(I)
107 00950 FOR J=2 TO I
108 00960 L3(J)=L3(J-1)/L2(J)
109 00970 L5(J)=L5(J-1)+X(J)/L3(J)
110 00980 NEXT J
111 00990 T1(I)=R0(I)*P(I)
112 01000 FOR J=2 TO I
113 01010 T1(J)=T1(J-1)*P(J)
114 01020 NEXT J
115 01030 T1(I+1)=1
116 01040 PRINT " "
117 01050 PRINT "I=";I
118 01060 PRINT " "
119 01070 T2(I)=R0(I)
120 01080 FOR J=2 TO I
121 01090 T2(J)=T1(J-1)
122 01100 NEXT J
123 01110 L3(I+1)=L3(I)
124 01120 X(I+1)=1-X(I)
125 01130 FOR J=1 TO I
WIDE LINE
126 01140 Z1=((T1(J+1)*M1-T1(J)*M1)*L3(J+1)/X(J+1)-(T1(J)*M1-T2(J)*M1)*L3(J)/X(J))/M1
WIDE LINE
127 01150 Z2=((T1(J+1)*M1-T1(J)*M1)*L3(J+1)/X(J+1)-(T1(J)*M1-T2(J)*M1)*L3(J)/X(J))/M1
128 01160 Q(J)=(Z1-G0*Z2)/(1+G0*M1/M1)
129 01170 S1(J)=Q(J)/T1(J)
130 01180 PRINT " J=";J;" Q=";Q(J);" S1=";S1(J);" L5=";L5(J);" T1/T9=";T1(J)
131 01190 NEXT J
132 01200 Q9=(1-T1(I)*M1-G0-G0*T1(I)*M1)*L3(I)/(X(I+1)*M1)
133 01210 Q9=Q9/(1+G0*M1/M1)
134 01220 Q0=(T1(I)*M1-R0(I)*M1-G0*T1(I)*M1-G0*R0(I)*M1)*L3(I)/(X(I)*M1)
135 01230 Q0=Q0/(1+G0*M1/M1)
136 01240 S0=Q0/R0(I)
137 01250 S2(I)=S0-Q9
138 01260 REM CALCULATING DATA TO GET SMIN
139 01270 D=(1-R0(I))/100
140 01280 FOR L=1 TO 101
141 01290 C(L)=R0(I)*D*(L-1)
142 01300 Y(L)=(C(L)*M0-G0*M1/M1)*C(L)*M0*0.5/C(L)
143 01310 NEXT L
144 01320 FOR J=1 TO I
145 01330 S2(I)=S2(I)+S1(J)
146 01340 NEXT J
147 01350 REM OBTAIN SMIN USING SIMPSON'S RULE
148 01360 H=Y(I)+Y(101)

```

```

140 01203 FOR K=2 TO 100
150 01204 IF K/2=INT(K/2) THEN 01207
151 01205 H=H+2*Y(K)
152 01206 GO TO 01208
153 01207 H=H+4*Y(K)
154 01208 NEXT K
155 01210 S0(I)=(D/3*H)**2/(1-C0*N1/M1)
156 01220 S3(I)=S2(I)/S0(I)
157 01230 S9(I)=(1-R0(I)*M1+C0-R0(I)*M1)*(1/R0(I)-1)/M1/(1-C0*N1/M1)
158 01240 S4(I)=S9(I)/S0(I)
159 01250 IF I=M THEN 01270
160 01260 P(I)=1/P1
161 01270 PRINT " "
162 01280 PRINT " P="P(I)," R="R(I)," X="X(I)," X1="X1(I)," S="S(I)
163 01290 PRINT " L2="L2(I)," L4="L4(I)
164 01291 PRINT " "
165 01292 PRINT " COLD WALL/HOT WALL TEMPERATURE RATIO, T0/T9="R0(I)
166 01293 PRINT " HEAT OUT AT COLD WALL="Q0," HEAT IN AT WARM WALL="Q9
167 01295 PRINT " ENTROPY PRODUCTION RATE AT COLD WALL="S0
168 01296 PRINT " ENTROPY PRODUCTION RATE AT WARM WALL="S9
169 01300 PRINT " MINIMUM ENTROPY PRODUCTION RATE, S0="S0(I)
170 01301 PRINT " ENTROPY PRODUCTION RATE FOR "I," SHIELDS, S2="S2(I)
171 01304 PRINT " MAXIMUM ENTROPY PRODUCTION RATE, S9="S9(I)
172 01310 PRINT " ENTROPY PRODUCTION RATE RATIOS, S3=S2/S0 AND S4=S9/S0"
173 01320 PRINT " S3="S3(I)," S4="S4(I)
174 01330 NEXT I
175 01340 GOTO 00170
176 01350 END
177 *EOP

```

NEWRAF

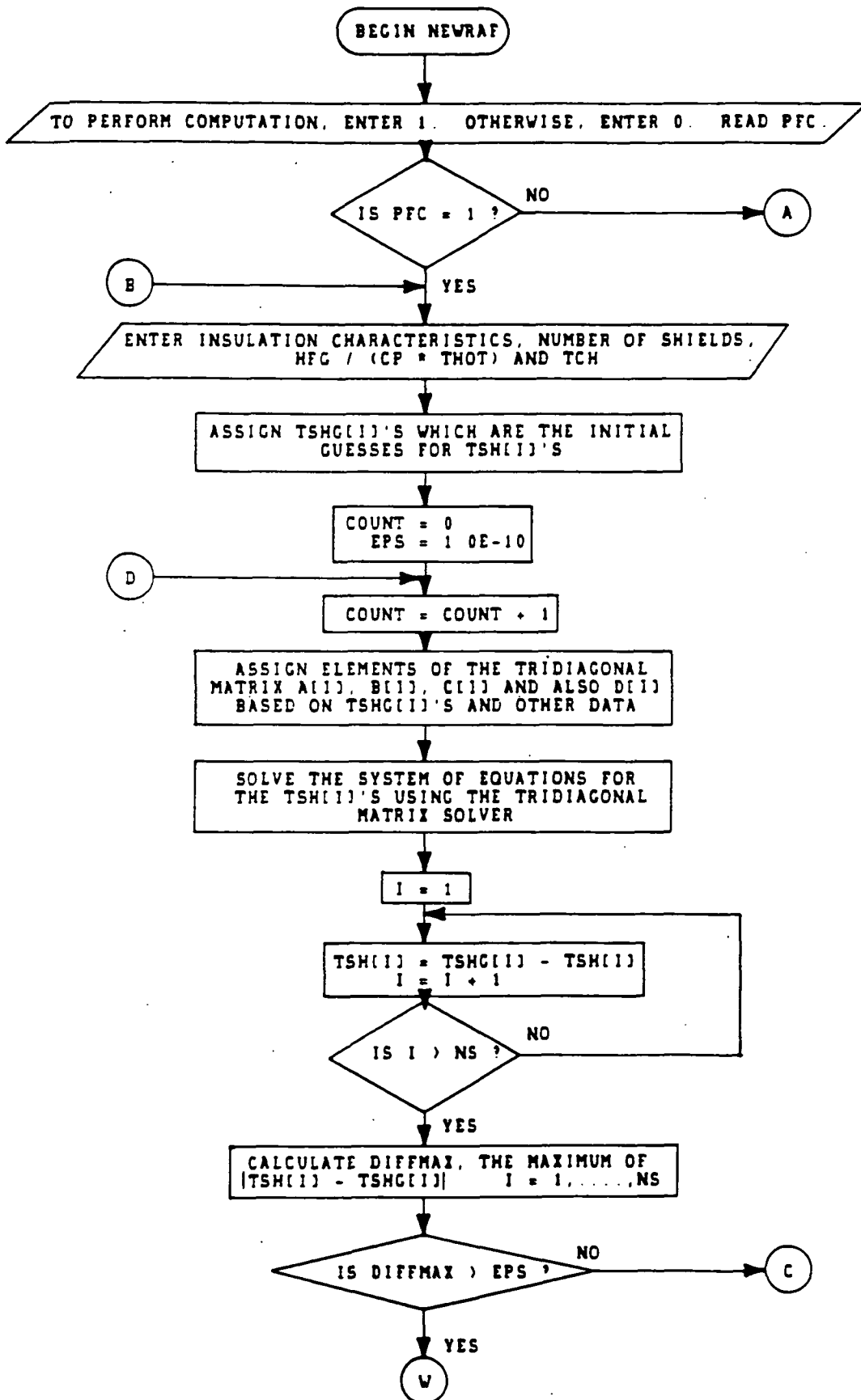
This program solves the original, complete, constrained optimization equations developed in Ref. [9] without the simplifying assumption suggested there which eliminated the dimensionless parameter, $h_{fg}/C_p T_H$. Only single-term thermal conductivity functions were considered in this analysis.

This program also recycles to the starting point. Consequently, the first input is either a 1, if a calculation is to be performed, or a 0, if no more work is to be done.

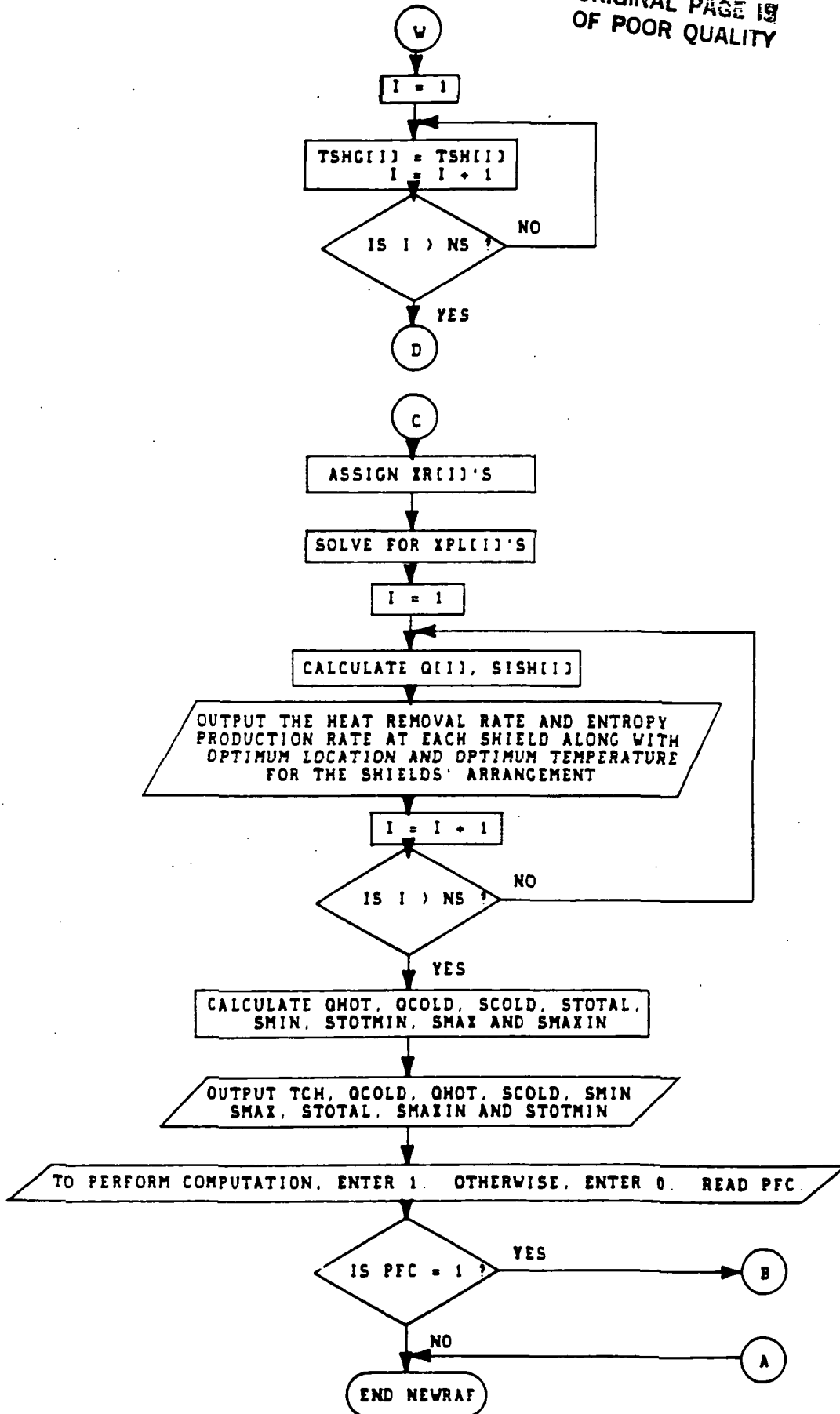
Next the program requests input of the insulation's characteristics, specifically, the exponent of temperature in the thermal conductivity function, the number of cooled shields, the dimensionless parameter $h_{fg}/C_p T_H$ for the boiloff from the insulated container, and $R = T_C/T_H$.

The output specifies the optimal characteristics of the given number of shields with the constraint that the cooling capacity is limited to the boiloff of the liquid due only to the heat leak through the insulation itself.

The flow chart and a program sample follows.



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PROGRAM NEVRAPH(INPUT,OUTPUT,AKM).

```

1
2
3
4
5 (.....)
6 (.....)
7 (.....)
8 (.....)
9 (.....)
10 (.....) NEVRAP (.....)
11 (.....)
12 (.....) J. C. CHATO & J. M. KHODADADI
13 (.....) DEPT OF MECHANICAL & INDUSTRIAL ENGRG.
14 (.....) UNIV OF ILLINOIS AT URBANA-CHAMPAIGN
15 (.....) 1206 W GREEN STREET
16 (.....) URBANA, IL 61801
17 (.....)
18 (.....) JULY 1983
19 (.....)
20 (.....)
21 (.....)
22 (.....)
23 (.....)
24 (.....)

```

```

25 (.....)
26 (* THIS PASCAL PROGRAM WAS DEVELOPED TO OPTIMIZE THE *)
27 (* LOCATION, TEMPERATURE AND HEAT DISSIPATION RATE *)
28 (* OF EACH COOLED SHIELD INSIDE AN INSULATION LAYER *)
29 (* THE THERMAL CONDUCTIVITY OF THE INSULATION HAS *)
30 (* THE GENERAL FORM, *)
31 (* *)
32 (* K = K1*(T**M) *)
33 (* *)
34 (* THE OBJECTIVE HAS BEEN TO SOLVE THE SET OF 2*NS-1 *)
35 (* NON-LINEAR EQUATIONS OBTAINED BY BEJAN, A. "DIS- *)
36 (* CRETE COOLING OF LOW HEAT LEAK SUPPORTS TO 4.2 K," *)
37 (* CRYOGENICS, VOL 15, 1975, PP.290-292. *)
38 (* *)
39 (* SOLUTION IS BASED ON THE NEWTON-RAPHSON TECHNIQUE *)
40 (* DISCUSSED BY STOECER, W. F., DESIGN OF THERMAL *)
41 (* SYSTEMS, 2ND EDITION, SECTION 6-11, PP 117-119, *)
42 (* MCGRAW-HILL BOOK CO., NY, 1980 *)
43 (* *)
44 (.....)
45 (.....)

```

LABEL LOG.

TYPE

```

51 ARRAYS=ARRAYS( 10) OF REAL. (* THE SIZE OF ARRAYS DETERMINES THE MAXIMUM NUMBER OF SHIELDS *)
52 ARRAYP=ARRAYS( 11) OF REAL. (* THE SIZE OF ARRAYP IS NS-1 *)
53 ARRAYT=ARRAYS( 20) OF REAL. (* THE SIZE OF ARRAYT SHOULD BE TWICE THE NUMBER OF SHIELDS *)

```

VAR

```

54
55
56 A ARRAYS, (* LOWER-DIAGONAL ELEMENTS OF THE TRIDIAGONAL MATRIX *)
57 B ARRAYS, (* DIAGONAL ELEMENTS OF THE TRIDIAGONAL MATRIX *)
58 C ARRAYS, (* UPPER-DIAGONAL ELEMENTS OF THE TRIDIAGONAL MATRIX *)
59 D ARRAYS, (* RIGHT-HAND SIDE OF THE SET OF EQUATIONS DURING ITERATIONS *)
60 Q I-TH DIMENSIONLESS HEAT REMOVAL RATE *)
61 Q1 ARRAYS, (* DIMENSIONLESS HEAT TRANSFER BETWEEN SHIELDS *)
62 S ARRAYS, (* DIMENSIONLESS ENTROPY PRODUCTION RATE FOR I-TH LAYER *)
63 SHAI REAL, (* MAXIMUM DIMENSIONLESS ENTROPY PRODUCTION RATE *)
64 SHMIN REAL, (* MINIMUM DIMENSIONLESS ENTROPY PRODUCTION RATE *)
65 STOTAL REAL, (* TOTAL DIMENSIONLESS ENTROPY PRODUCTION RATE *)
66 STOTMIN REAL, (* STOTAL / SHMIN *)
67 SISH ARRAYS, (* I-TH DIMENSIONLESS ENTROPY PRODUCTION RATE *)
68 TSH ARRAYS, (* I-TH SHIELD / HOT WALL TEMPERATURE RATIO, ALWAYS ( 1. *)
69 TSHG ARRAYS, (* GUESSED I-TH SHIELD / HOT WALL TEMPERATURE RATIO, ALWAYS ( 1. *)
70 WORK ARRAYS, (* DUMMY VARIABLES *)
71 I ARRAYS, (* SPACING BETWEEN NEIGHBORING SHIELDS / INSULATION THICKNESS *)
72 IPI ARRAYS, (* DISTANCE FROM COLD WALL / INSULATION THICKNESS *)
73 IR ARRAYS, (* X(I) / X(I-1) *)
74
75
76
77
78 AKM TEXT, (* OUTPUT FILE TO BE USED IF DESIRED *)
79 BETA REAL, (* PARAMETER DEFINED IN PROCEDURE INPUT *)

```



```

80      BOLD          REAL          (* DUMMY VARIABLE USED IN SOLVING THE TRIDIAGONAL MATRIX *)
81      COUNT        INTEGER       (* NUMBER OF ITERATIONS NEEDED TO DETERMINE TSH(I)'S *)
82      DELTATC, DEN  REAL          (* DUMMY VARIABLES *)
83      DIFF, DIFFMAX REAL          (* DUMMY VARIABLES USED IN CHECKING CONVERGENCE *)
84      DIW, DMAX, DMIN REAL        (* DUMMY VARIABLES USED IN SOLVING THE TRIDIAGONAL MATRIX *)
85      EPS           REAL          (* A SMALL VALUE USED TO OBSERVE IF CONVERGENCE IS OBTAINED *)
86      GI, GIM1, GIP1 REAL        (* DUMMY VARIABLES *)
87      GOLD         REAL          (* DUMMY VARIABLE USED IN SOLVING THE TRIDIAGONAL MATRIX *)
88      I, J          INTEGER       (* INDICES FOR LOOPS *)
89      ITERIN       INTEGER       (* INDEX USED TO TERMINATE ITERATIONS *)
90      M            REAL          (* POWER OF THE THERMAL CONDUCTIVITY EQUATION *)
91      MM1          REAL          (* EQUALS M-1 *)
92      MP1          REAL          (* EQUALS M+1 *)
93      NS           INTEGER       (* NUMBER OF SHIELDS *)
94      NSP1         INTEGER       (* EQUALS NS-1 *)
95      PFC          INTEGER       (* PROGRAM FLOW CONTROLLER *)
96      TCH          REAL          (* COLD WALL / HOT WALL TEMPERATURE RATIO, ALWAYS ( 1 *)
97      TI, TMC      REAL          (* DUMMY VARIABLES *)
98      QCOLD        REAL          (* HEAT OUT AT COLD WALL *)
99      QHOT         REAL          (* HEAT IN AT HOT WALL *)
00      SCOLD        REAL          (* ENTROPY PRODUCTION RATE AT COLD WALL *)
01      XTOTAL       REAL          (* SUM OF X(I)'S; SHOULD EQUAL 1 AFTER SUCCESSFUL COMPUTATION *)
02
03
04
05

```

```

06  PROCEDURE INPUTH.
07  BEGIN
08  WRITELN
09  WRITELN( 'ENTER ---- M NS BETA TCH (----)',
10  WRITELN( ' )' )
11  WRITELN( 'WHERE M ---- POWER IN THE THERMAL CONDUCTIVITY EQUATION',
12  WRITELN( 'NS ---- NUMBER OF SHIELDS',
13  WRITELN( 'BETA -- HFC / (CP*THOT)',
14  WRITELN( 'HFC --- HEAT OF VAPORIZATION (J/KG)',
15  WRITELN( 'CP ---- SPECIFIC HEAT AT CONSTANT PRESSURE (J/KG K)',
16  WRITELN( 'THOT -- HOT WALL TEMPERATURE (K)',
17  WRITELN( 'TCH --- COLD WALL / HOT WALL TEMPERATURE RATIO, ALWAYS ( 1',
18  WRITELN( ' )' )
19  END
20  (* INPUT OF DATA HEADING *)

```

```

23  PROCEDURE PFC.
24  BEGIN
25  WRITELN
26  WRITELN( 'TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0',
27  WRITELN( ' )' )
28  END
29  (* PFC *)

```

```

32  PROCEDURE SINGLESPACE.
33  BEGIN
34  WRITELN( ' )' )
35  END
36  (* SINGLE SPACE IN OUTPUT *)

```

```

39  FUNCTION PWR(XI, E REAL) REAL.
40  VAR
41  A
42  BEGIN
43  A = E*LN(XI)
44  PWR = EXP(A)
45  END
46  (* COMPUTE XI**E *)

```

```

49  FUNCTION MAXOF2(N01, N02 REAL) REAL.
50  BEGIN
51  IF N01 < N02 THEN
52  IF N01 > N02 THEN
53  MAXOF2 = N01
54  ELSE
55  MAXOF2 = N02
56  ELSE
57  MAXOF2 = N01
58  END
59  (* DETERMINES THE LARGEST OF THE TWO GIVEN NUMBERS *)

```

```

60
61
62 FUNCTION MINOF2(NO1,NO2,REAL):REAL;
63 BEGIN      (* DETERMINES THE SMALLEST OF THE TWO GIVEN NUMBERS *)
64   IF NO1<NO2 THEN
65     IF NO1<NO2 THEN
66       MINOF2:=NO2
67     ELSE
68       MINOF2:=NO1
69     ELSE
70       MINOF2:=NO1;
71 END;      (* DETERMINES THE SMALLEST OF THE TWO GIVEN NUMBERS *)
72
73
74
75
76
77
78
79      (* MAIN PROGRAM BODY *)
80
81 BEGIN
82   PFC=1;
83   READLN;
84   READ(PFC);
85   WHILE PFC<= 10 DO
86     BEGIN
87
88       (* THIS BLOCK IS USED TO INPUT THE INSULATION THERMAL CONDUCTIVITY, NUMBER *)
89       (* OF SHEETS, HFC/(CP*THOT) AND COLD WALL / HOT WALL TEMPERATURE RATIO *)
90
91       INPUT
92       READLN
93       READ(M,NS,BETA,TCH);
94       SINGLESPACE;
95       WRITELN('      THERMAL CONDUCTIVITY OF THE INSULATION IS K = K1*T**',M 3-1),
96       WRITELN('      HFC / (CP*THOT) = ',BETA,9 5);
97       SINGLESPACE;
98       SINGLESPACE;
99
100      MP:=M-1 0;
101      MM:=M-1 0;
102
103      (* INITIAL GUESSED VALUES FOR TSHG(I)'S ARE ENTERED *)
104
105      DELTATG:=(1 0-TCH)/(NS+1 0);
106      FOR J:=1 TO NS DO TSHG(J):=J*DELTATG+TCH;
107
108      (* VARIABLE USED TO CHECK CONVERGENCE CRITERION IS SET AND THE ITERATIVE PROCEDURE *)
109      (* OF NEWTON-RAPHSON METHOD IS STARTED *)
110
111      EPS:=1 0E-10;
112      COUNT:=0;
113      ITERIN:=0;
114      REPEAT
115        COUNT:=COUNT+1;
116        FOR I:=1 TO NS DO
117          BEGIN
118            GI:=TSHG(I);
119            IF NS=1 THEN
120              IF I<0 THEN
121                IF I<0 THEN
122                  BEGIN
123                    GI:=TSHG(I-1);
124                    GIP:=TSHG(I+1);
125                  END
126                ELSE
127                  BEGIN
128                    GI:=TSHG(I-1);
129                    GIP:=1 0;
130                  END
131              ELSE
132                BEGIN
133                  GI:=TCH;
134                  GIP:=TSHG(I+1);
135                END
136            END
137          ELSE
138            BEGIN
139              GI:=TCH;
140            END
141          END
142
143          (* ... (the rest of the code is partially obscured by the image quality) ... *)

```

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```

40      GIP1 =1.0
41      END.
42
43      (* ELEMENTS OF THE TRIAGONAL MATRIX ARE COMPUTED *)
44
45      A(1) =PWR(GIP1,MP1)-PWR(GI,M)*(-M*GI*MP1*(TCH-BETA));
46      B(1) =MPI*PWR(GI,MM1)*(M*GIM*(BETA-TCH+GI)+GI*(-2.0*(BETA-TCH)));
47      C(1) =B(1)+MPI*PWR(GI,MM1)*GI*(-M*(BETA-TCH)-GI*(M-2.0));
48      C(1) =MPI*PWR(GIP1,M)*(BETA-TCH+GIM);
49      D(1) =GIM*(PWR(GIP1,MP1)-PWR(GI,MP1))+BETA*MPI*PWR(GI,M)-TCH*MP1*PWR(GI,M)+MPI*PWR(GI,MP1);
50      D(1) =D(1)-GI*(PWR(GI,M)*(TCH-BETA)-BETA*MPI*PWR(GI,M)+TCH*MP1*PWR(GI,M)-MPI*PWR(GI,MP1));
51      D(1) =D(1)+GIP1*(BETA*PWR(GIP1,M)-TCH*PWR(GIP1,M))
52      END.
53      A(1) =0.0.
54      C(1) =0.0.
55
56      (* THE TRIAGONAL MATRIX SOLVER IS SHOWN IN THIS BLOCK *)
57      (* SEE WESTLAKE, J. R. , A HANDBOOK OF NUMERICAL MATRIX *)
58      (* INVERSION AND SOLUTION OF LINEAR EQUATIONS, SECTION *)
59      (* 2.7, PP. 34-35, JOHN WILEY & SONS, INC , NY, 1968 *)
60
61      IF B(1)=0.0 THEN GOTO 100.
62      BOLD =B(1)/B(1);
63      GOLD =D(1)/B(1);
64      WORK(1) =GOLD
65      WORK(NS-1) =BOLD.
66      DMAX =ABS(B(1));
67      DMIN =ABS(B(1));
68      FOR I =2 TO NS DO
69      BEGIN
70      DIW =B(I)-A(I)*BOLD.
71      IF DIW=0.0 THEN GOTO 100.
72      DMAX =MAXOF2(DMAX,ABS(DIW));
73      DMIN =MINOF2(DMIN,ABS(DIW));
74      GOLD =(D(I)-A(I)*GOLD)/DIW.
75      WORK(I) =GOLD.
76      BOLD =C(I)/DIW.
77      WORK(NS) =BOLD
78      END
79      TSH(NS) =GOLD
80      I =NS
81      FOR I =0 TO NS DO
82      BEGIN
83      J =0.
84      GOLD =WORK(I)-WORK(I)+NS*GOLD.
85      TSH(I) =GOLD
86      END.
87
88      (* NEWLY CALCULATED VALUES OF TSH(I)'S ARE COMPUTED *)
89
90      FOR I =1 TO NS DO TSH(I) =TSH(I)-TSH(I).
91
92      (* CONVERGENCE IS CHECKED. IF THE CRITERION IS SATISFIED, THE ITERATION IS *)
93      (* TERMINATED, OTHERWISE THE NEWLY CALCULATED TSH(I)'S ARE USED AS NEW *)
94      (* GUESSES FOR ANOTHER ROUND OF ITERATION *)
95
96      DIFFMAX =1.0E-15.
97      FOR I =1 TO NS DO
98      BEGIN
99      DIFF =ABS(TSH(I)-TSH(I)).
100     DIFFMAX =MAXOF2(DIFF,DIFFMAX)
101     END.
102     IF DIFFMAX<=EPS THEN
103     ITERIN =1
104     ELSE
105     FOR I =1 TO NS DO TSH(I) =TSH(I).
106     UNTIL ITERIN=1.
107
108     (* IN THIS BLOCK QUANTITIES USED IN DETERMINING THE SHIELDS' SPACINGS ARE COMPUTED *)
109
110     FOR I =1 TO NS DO
111     BEGIN
112     T =TSH(I).
113     IF NS(I) THEN
114     IF I(1) THEN
115     IF I(ONS) THEN
116     T(I) =TSH(I-1)
117     ELSE
118     T(I) =TSH(I-1)
119     ELSE

```

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```

20     TIME = TCH
21     ELSE
22     TIME = TCH;
23     XRI(1) = (MP1 * PWR(TI, M) * (TI - TIME)) / (PWR(TI, MP1) - PWR(TIME, MP1));
24     END.
25     DEN = 1.0;
26     FOR I = 1 TO NS DO DEN = DEN * XRI(NS - I + 1) + 1.0;
27     NSPI = NS + 1;
28
29     (* FINALLY, SPACINGS BETWEEN SHIELDS AND OTHER QUANTITIES OF INTEREST ARE CALCULATED *)
30
31     X(1) = 1.0 / DEN;
32     XPL(1) = X(1);
33     XTOTAL = X(1);
34     FOR I = 2 TO NSPI DO
35     BEGIN
36     X(I) = X(I-1) * XRI(I-1);
37     IF I < NSPI THEN XPL(I) = XPL(I-1) * X(I);
38     XTOTAL = XTOTAL + X(I)
39     END.
40     IF (ABS(XTOTAL - 1.0) > 1.0E-5) THEN GOTO 100;
41     QHS(1) = (PWR(TSH(1), MP1) - PWR(TCH, MP1)) / (X(1) * MP1);
42     QHS(NSPI) = (1.0 - PWR(TSH(NS), MP1)) / (X(NSPI) * MP1);
43     FOR I = 2 TO NS DO QHS(I) = (PWR(TSH(I), MP1) - PWR(TSH(I-1), MP1)) / (X(I) * MP1);
44     SINGLESPACE;
45     WRITELN:      NUMBER OF SHIELDS      = ', NS 2);
46     WRITELN:      NUMBER OF ITERATIONS = ', COUNT 2);
47     SINGLESPACE
48     SINGLESPACE;
49     WRITELN:
50     WRITELN:      HEAT REMOVAL          ENTROPY PRODUCTION          OPTIMUM
51     WRITELN:      RATE                  RATE                  LOCATION
52     WRITELN:      -----              -----              -----
53     FOR I = 1 TO NS DO
54     BEGIN
55     QHS(I) = QHS(I) - QHS(1);
56     SISH(I) = QHS(I) / TSH(I);
57     WRITELN:      SHIELD ', I, 2, ' ', S, QHS(I) - 9.5, ' ', S, SISH(I) - 9.5, ' ', S, XPL(I) - 9.5, ' ', S, TSH(I) - 9.5);
58     END;
59     SINGLESPACE;
60     SINGLESPACE;
61     QHOT = QHS(NSPI);
62     QCOLD = QHS(1);
63     SCOLD = QCOLD / TCH;
64     STOTAL = SCOLD - QHOT;
65     FOR J = 1 TO NS DO STOTAL = STOTAL + SISH(J);
66     IF M < 0 THEN
67     SMIN = SQRT((1.0 - PWR(TCH, (M/2.0))) / (M/2.0))
68     ELSE
69     SMIN = SQRT((1.0 / TCH));
70     STOTMIN = STOTAL / SMIN;
71     SMAX = (1.0 - PWR(TCH, MP1)) * (1.0 / TCH - 1.0 / MP1);
72     SMAXIN = SMAX / SMIN;
73     SINGLESPACE;
74     WRITELN:      COLD WALL / HOT WALL TEMPERATURE RATIO = ', TCH 14.6);
75     WRITELN:      HEAT OUT AT COLD WALL = ', QCOLD 14.6);
76     WRITELN:      HEAT IN AT HOT WALL = ', QHOT 14.6);
77     WRITELN:      ENTROPY PRODUCTION RATE AT COLD WALL = ', SCOLD 14.6);
78     WRITELN:      ENTROPY PRODUCTION RATE AT HOT WALL = ', -QHOT 14.6);
79     WRITELN:      MINIMUM ENTROPY PRODUCTION RATE = ', SMIN 14.6);
80     WRITELN:      MAXIMUM ENTROPY PRODUCTION RATE = ', SMAX 14.6);
81     WRITELN:      TOTAL ENTROPY PROD. RATE WITH ', NS 2, ' SHIELDS = ', STOTAL 14.6);
82     WRITELN:      MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO = ', SMAXIN 14.6);
83     WRITELN:      TOTAL / MINIMUM ENTROPY PRODUCTION RATIO = ', STOTMIN 14.6);
84     100 SINGLESPACE;
85     IF (DIM = 0.0) OR (B(1) = 0.0) THEN
86     BEGIN
87     SINGLESPACE;
88     WRITELN:      ---) CHECK THE ASSEMBLY OF COEFFICIENTS TO BE USED IN TRIDIAGONAL MATRIX (---);
89     WRITELN:      ---) CHECK THE TRIDIAGONAL MATRIX SOLVER (---);
90     END.
91     IF (ABS(XTOTAL - 1.0) > 1.0E-5) THEN
92     BEGIN
93     SINGLESPACE;
94     WRITELN:      ---) XTOTAL IS NOT EQUAL TO 1.0 (---);
95     WRITELN:      ---) COMPUTATIONS ARE NOT CORRECT (---);
96     END.
97     SINGLESPACE;
98     SINGLESPACE;
99

```

CONTINUED FROM PAGE 74

00 PFCH.
01 READLN.
02 READ(PFC)
03 END
04 ENC
05 /EOP.

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 1

ENTER ----> M NS BETA TCH <----

WHERE: M ----- POWER IN THE THERMAL CONDUCTIVITY EQUATION
 NS ---- NUMBER OF SHIELDS
 BETA -- HFG / (CP*THOT)
 HFG --- HEAT OF VAPORIZATION [J/KG]
 CP ---- SPECIFIC HEAT AT CONSTANT PRESSURE [J/KG K]
 THOT -- HOT WALL TEMPERATURE [K]
 TCH --- COLD WALL / HOT WALL TEMPERATURE RATIO, ALWAYS : 1

? 1

1.0 3 0.0145 0.001

THERMAL CONDUCTIVITY OF THE INSULATION IS $K = K1*T**1.0$
 $HFG / (CP*THOT) = 0.01450$

NUMBER OF SHIELDS = 3
 NUMBER OF ITERATIONS = 9

	HEAT REMOVAL RATE	ENTROPY PRODUCTION RATE	OPTIMUM LOCATION	OPTIMUM TEMPERATURE
	-----	-----	-----	-----
SHIELD 1	0.10438	1.56143	0.09719	0.06685
SHIELD 2	0.25983	1.12595	0.28870	0.23076
SHIELD 3	0.47781	0.89782	0.58568	0.53219

COLD WALL / HOT WALL TEMPERATURE RATIO = 0.001000
 HEAT OUT AT COLD WALL = 0.022985
 HEAT IN AT HOT WALL = 0.864998
 ENTROPY PRODUCTION RATE AT COLD WALL = 22.984544
 ENTROPY PRODUCTION RATE AT HOT WALL = -0.864998
 MINIMUM ENTROPY PRODUCTION RATE = 3.751018
 MAXIMUM ENTROPY PRODUCTION RATE = 499.499501
 TOTAL ENTROPY PROD. RATE WITH 3 SHIELDS = 25.704743
 MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO = 133.163725
 TOTAL / MINIMUM ENTROPY PRODUCTION RATIO = 6.852738

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 1

ENTER ----> M NS BETA TCH <----

WHERE: M ----- POWER IN THE THERMAL CONDUCTIVITY EQUATION
 NS ---- NUMBER OF SHIELDS
 BETA -- HFG / (CP*THOT)
 HFG --- HEAT OF VAPORIZATION [J/KG]
 CP ---- SPECIFIC HEAT AT CONSTANT PRESSURE [J/KG K]
 THOT -- HOT WALL TEMPERATURE [K]
 TCH --- COLD WALL / HOT WALL TEMPERATURE RATIO, ALWAYS : 1

? 1

1.0 2 0.0154 0.000806

THERMAL CONDUCTIVITY OF THE INSULATION IS $K = K1 * T ** 1.0$
 $MFG / (CP * THOT) = 0.01540$

NUMBER OF SHIELDS = 2
 NUMBER OF ITERATIONS = 8

	HEAT REMOVAL RATE	ENTROPY PRODUCTION RATE	OPTIMUM LOCATION	OPTIMUM TEMPERATURE
SHIELD 1	0.19732	1.97595	0.16252	0.09986
SHIELD 2	0.59037	1.48999	0.48495	0.39623

COLD WALL / HOT WALL TEMPERATURE RATIO = 0.000806
 HEAT OUT AT COLD WALL = 0.030677
 HEAT IN AT HOT WALL = 0.818366
 ENTROPY PRODUCTION RATE AT COLD WALL = 38.061092
 ENTROPY PRODUCTION RATE AT HOT WALL = -0.818366
 MINIMUM ENTROPY PRODUCTION RATE = 3.776103
 MAXIMUM ENTROPY PRODUCTION RATE = 619.846992
 TOTAL ENTROPY PROD. RATE WITH 2 SHIELDS = 40.708665
 MAXIMUM / MINIMUM ENTROPY PRODUCTION RATIO = 164.149921
 TOTAL / MINIMUM ENTROPY PRODUCTION RATIO = 10.780603

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 0

0.072 CP SECS, 11471B CM USED.

/BYE:

3KMUFTC COSTS: 255.028 SRUS AT 0.0059 = \$1.50

DESINS

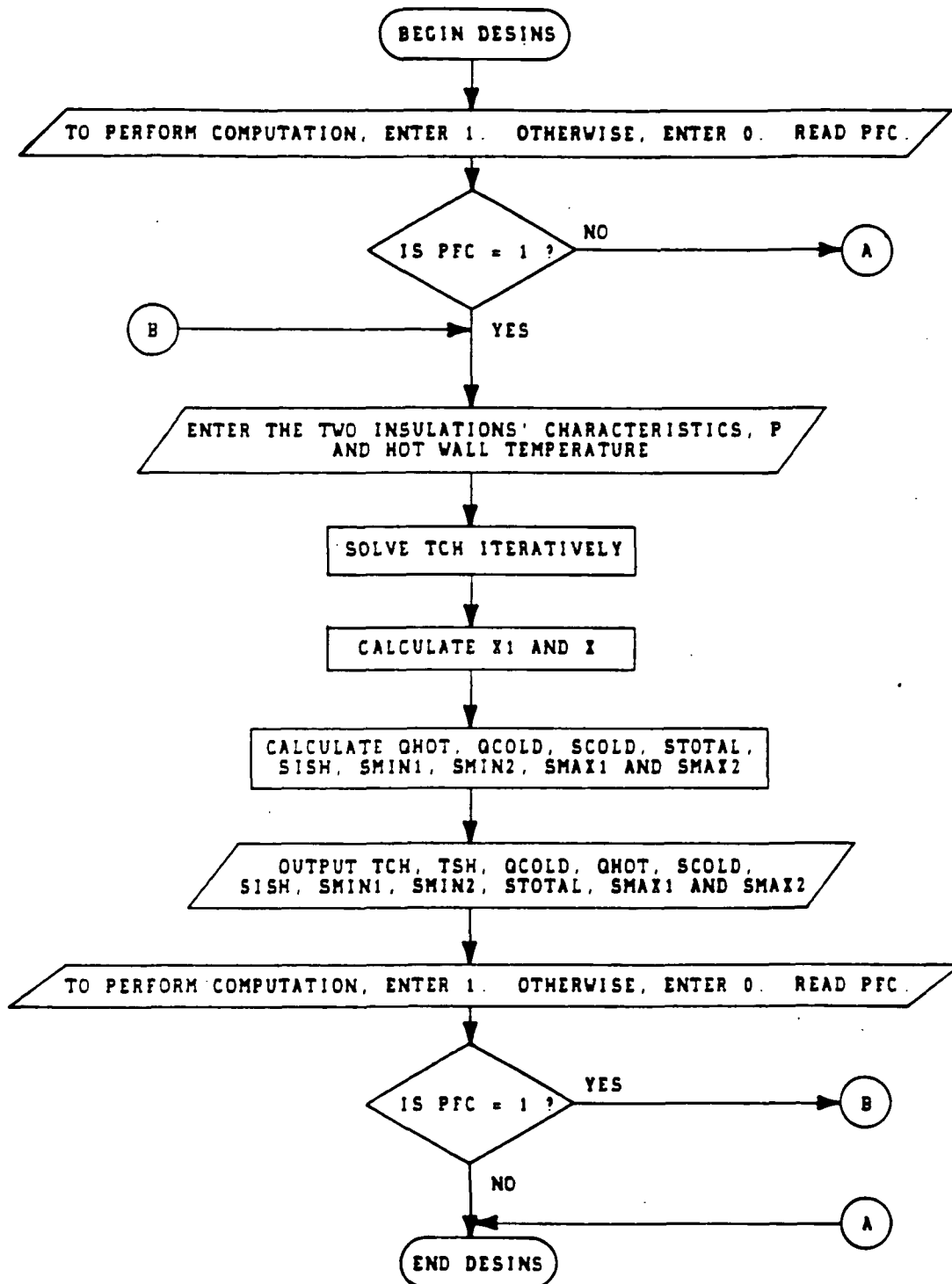
This program optimizes the characteristics of a single cooled shield with different insulations on the two sides. Only one-term thermal conductivity functions are considered.

This program also recycles to the starting point; thus the first input is 1, if a calculation is to be performed, or 0 if no more work is to be done.

Next inputs are the characteristics of the two insulations, specifically, the exponents of temperature in the thermal conductivity functions on the hot and cold sides of the shield, a coefficient ratio ALFA (defined in the program), the shield to cold wall temperature ratio, $P = T_S/T_C$, and the hot wall temperature, T_H .

The output specifies the optimal characteristics of the cooled shield as well as other, related information.

The flow diagram and a program sample follows.

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```

80
81
82 PROCEDURE INPUTH;
83 BEGIN      (* INPUT OF DATA HEADING *)
84   WRITELN;
85   WRITELN(' ENTER ----) M N ALFA P THOT (----)');
86   WRITELN(' ');
87   WRITELN(' WHERE M ---- POWER OF THE THERMAL CONDUCTIVITY EQUATION ON THE HOT SIDE');
88   WRITELN(' N ---- POWER OF THE THERMAL CONDUCTIVITY EQUATION ON THE COLD SIDE');
89   WRITELN(' ALFA -- (K2*(M+1))/(K1*(N+1))');
90   WRITELN(' P ---- SHIELD / COLD WALL TEMPERATURE RATIO, ALWAYS ) 1');
91   WRITELN(' THOT -- HOT WALL TEMPERATURE (K)');
92   WRITELN(' ');
93 END;
94
95
96
97 PROCEDURE PFCM;
98 BEGIN      (* PFCM *)
99   WRITELN;
100  WRITELN(' TO PERFORM COMPUTATION, ENTER 1 OTHERWISE, ENTER 0 ');
101  WRITELN;
102 END;
103
104
105
106 PROCEDURE SINGLESPEAC;
107 BEGIN      (* SINGLE SPACE IN OUTPUT *)
108   WRITELN(' ');
109 END;
110
111
112
113 FUNCTION PWR(X:REAL) REAL;
114 VAR
115   A      REAL;
116 BEGIN   (* COMPUTE X**E *)
117   A :=E*LN(X);
118   PWR :=EXP(A);
119 END;
120
121
122
123 FUNCTION D(E:REAL) REAL;
124 BEGIN   (* FUNCTIONAL D *)
125   D :=(E-1.0)*PWR(X,E)-E/(PWR(X,(1.0-E)))-(1.0/SQR(X));
126 END;
127
128
129
130 FUNCTION F(E:REAL) REAL;
131 BEGIN   (* FUNCTIONAL F *)
132   F :=(PWR(X,(E-1.0)))-PWR(X,E)-1.0*(1.0/X);
133 END;
134
135
136
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```

* MAIN PROGRAM BODY *

```

BEGIN
PFCM;
REALM;
READ:PFC;
WHILE PFC=1 DO
BEGIN
(* THIS BLOCK IS USED TO INPUT THE TWO INSULATION THERMAL CONDUCTIVITIES, *)
(* SHIELD / COLD WALL TEMPERATURE RATIO AND HOT WALL TEMPERATURE *)

INPUTH;
READLN;
READ(M,N,ALFA,P,THOT);
SINGLESPEAC;
WRITELN(' THERMAL CONDUCTIVITY OF THE INSULATION ON THE HOT SIDE IS K = K1*(T**M)');
WRITELN(' THERMAL CONDUCTIVITY OF THE INSULATION ON THE COLD SIDE IS K = K2*(T**N)');
WRITELN(' (K2*(M+1))/(K1*(N+1)) = ',ALFA);

```

160 WRITELN(''
 161 SINGLESPEACE,
 162 SINGLESPEACE,
 163
 164

165 MPI.=M+1.0;
 166 MPI.=M+1.0,
 167 TCH =0 000001,
 168 CC =0 1;
 169 DD =1 0,
 170 COUNT =0,
 171

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(* THIS BLOCK CALCULATES TCH ITERATIVELY *)

REPEAT

175 TSH =P*TCH,
 176 C =D(N,P)*D(N,P)/F(N,P)-PWR(TCH,(2.0-N))*D(M,TSH)*D(M,TSH)/F(M,TSH)/ALFA,
 177 G1 =C*DD,
 178 IF G1<0.0 THEN GOTO 100,
 179 IF G1=0.0 THEN GOTO 200,
 180 CC =(-0.1)*CC,
 181 IF ABS(CC)<0.000001 THEN GOTO 200,
 182 DD =-DD,
 183 100 TCH =TCH+CC,
 184 IF 'TCH<0.999999' OR 'TCH<0.000001' THEN
 185 BEGIN
 186 TCH =TCH-0.9*CC,
 187 CC =0.1*CC,
 188 IF ABS(CC)<0.000001 THEN IND =1
 189 END,
 190 200 COUNT =COUNT+1,
 191 UNTIL 'G1=0.0' OR 'ABS(CC)<0.000001' OR 'IND=1',
 192
 193

IF IND=1 THEN

195 BEGIN
 196 SINGLESPEACE,
 197 SINGLESPEACE,
 198 SINGLESPEACE,
 199 WRITELN(' ---) OPTIMUM CRITERION CANNOT BE SATISFIED (---)),
 200 WRITELN(' ---) USE SINGLE INSULATION WITH THE LOWER CONDUCTIVITY (---)),
 201 GOTO 300
 202 END
 203

(* OTHER QUANTITIES OF INTEREST ARE COMPUTED IN THIS SECTION *)

206 X1 =-ALFA*PWR(TCH,(N-1.0))*D(N,P)/D(M,TSH),
 207 X =X1/(1.0+X1),
 208 QHOT =(1.0-PWR(TSH,MPI))/((1.0-X)*MPI),
 209 QCOLD =ALFA*PWR(TCH,MPI)*(PWR(P,MPI)-1.0)/(PWR(THOT,(M-N))*X*MPI),
 210 SCOLD =QCOLD/TCH,
 211 STOTAL =(F(M,TSH)/(1.0-X)+ALFA*PWR(TCH,M)*F(N,P)/X)/MPI,
 212 SISH = QHOT-QCOLD/TSH,
 213 IF M=0.0 THEN
 214 SMIN1 =SOR(LN(1.0/TCH))
 215 ELSE
 216 SMIN1 =SOR((1.0-PWR(TCH,(M/2.0)))/(M/2.0)),
 217 IF N=0.0 THEN
 218 SMIN2 =SOR(LN(1.0/TCH))
 219 ELSE
 220 SMIN2 =SOR((1.0-PWR(TCH,(M/2.0)))/(M/2.0)),
 221 SMAX1 =F(M,TCH)/MPI,
 222 SMAX2 =F(N,TCH)/MPI,
 223
 224

SINGLESPEACE,

226 WRITELN(' NUMBER OF ITERATIONS = ',COUNT 0),
 227 WRITELN(' COLD WALL / HOT WALL TEMPERATURE RATIO = ',TCH 14.6),
 228 WRITELN(' SHIELD / HOT WALL TEMPERATURE RATIO = ',TSH 14.6),
 229 WRITELN(' SHIELD LOCATION = ',X 14.6),
 230 WRITELN(' HEAT OUT AT SHIELD = ',QHOT-QCOLD 14.6),
 231 WRITELN(' HEAT OUT AT COLD WALL = ',QCOLD 14.6),
 232 WRITELN(' HEAT IN AT HOT WALL = ',QHOT 14.6),
 233 WRITELN(' ENTROPY PRODUCTION RATE AT COLD WALL = ',SCOLD 14.6),
 234 WRITELN(' ENTROPY PRODUCTION RATE AT HOT WALL = ',-QHOT 14.6),
 235 WRITELN(' ENTROPY PRODUCTION RATE AT SHIELD = ',SISH 14.6),
 236 WRITELN(' MINIMUM ENTROPY PRODUCTION RATE BASED ON K1*T**N = ',SMIN1 14.6),
 237 WRITELN(' MINIMUM ENTROPY PRODUCTION RATE BASED ON K2*T**N = ',SMIN2 14.6),
 238 WRITELN(' TOTAL ENTROPY PRODUCTION RATE = ',STOTAL 14.6),
 239 WRITELN(' ENTROPY PROD. W/O SHIELD BASED ON K1*T**N = ',SMAX1 14.6),

240
241
242
243
244
245
246
247
248

WRITE(1
300 SINGLES
SINGLES
SINGLES
PFCH,
READEN,
READ(PFC)
END
END

ENTROPY PROD W/O SHIELD BASED ON K2-T**M

= ' ,SHAT2 14 6).

83

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TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 1

ENTER ----> M N ALFA P THOT <----

WHERE: M ----- POWER OF THE THERMAL CONDUCTIVITY EQUATION ON THE HOT SIDE
 N ----- POWER OF THE THERMAL CONDUCTIVITY EQUATION ON THE COLD SIDE
 ALFA -- $[K2*(M+1)]/[K1*(N+1)]$
 P ----- SHIELD / COLD WALL TEMPERATURE RATIO, ALWAYS 1
 THOT -- HOT WALL TEMPERATURE [K]

? 1.0 0.0 20.0 4.5 300.0

THERMAL CONDUCTIVITY OF THE INSULATION ON THE HOT SIDE IS $K = K1*(T**1.0)$.
 THERMAL CONDUCTIVITY OF THE INSULATION ON THE COLD SIDE IS $K = K2*(T**0.0)$.
 $[K2*(M+1)]/[K1*(N+1)] = 20.00$
 HOT WALL TEMPERATURE = 300.00 [K]

NUMBER OF ITERATIONS	=	36
COLD WALL / HOT WALL TEMPERATURE RATIO	=	0.001666
SHIELD / HOT WALL TEMPERATURE RATIO	=	0.007497
SHIELD LOCATION	=	0.390755
HEAT OUT AT SHIELD	=	0.820144
HEAT OUT AT COLD WALL	=	0.000497
HEAT IN AT HOT WALL	=	0.820641
ENTROPY PRODUCTION RATE AT COLD WALL	=	0.298568
ENTROPY PRODUCTION RATE AT HOT WALL	=	-0.820641
ENTROPY PRODUCTION RATE AT SHIELD	=	109.396253
MINIMUM ENTROPY PRODUCTION RATE BASED ON $K1*T**M$	=	3.680131
MINIMUM ENTROPY PRODUCTION RATE BASED ON $K2*T**N$	=	40.925828
TOTAL ENTROPY PRODUCTION RATE	=	178.307751
ENTROPY PROD. W/O SHIELD BASED ON $K1*T**M$	=	299.619216
ENTROPY PROD. W/O SHIELD BASED ON $K2*T**N$	=	598.241762

TO PERFORM COMPUTATION, ENTER 1. OTHERWISE, ENTER 0.

? 0

0.044 CP SECS, 10233B CM USED.

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16. Abstract A relatively simple method has been developed to optimize the location, temperature, and heat dissipation rate of each cooled shield inside an insulation layer. The method is based on the minimization of the entropy production rate which is proportional to the heat leak across the insulation. The results show that the maximum number of shields to be used in most practical applications is three. However, cooled shields are useful only at low values of the overall, cold wall to hot wall absolute temperature ratio. The performance of the insulation system is relatively insensitive to deviations from the optimum values of the temperature and location of the cooling shields. Design curves are presented for rapid estimates of the locations and temperatures of cooling shields in various types of insulations, and an equation is given for calculating the cooling loads for the shields.			
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