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Rocket Thrust Chamber Thermal Barrier Coatings

Final Report under contract NAS3-23262

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Rocket Thrust Chamber Thermal Barrier Coatings

Final Report under contract NAS3-23262

by A.P. Batakis J.W. Vogan

for National Aeronautics & Space Administration Communications & Propulsion Center 21000 Brookpark Road Cleveland, OH 44135

July 1985 SR84-R-5052-16



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1

INTRODUCTION

The objective of this program is to develop a zirconia based thermal barrier coating (TBC) system for use in high heat flux rocket applications such as the Space Shuttle main engine (SSME) thrust chamber. The technology developed is also pertinent to other applications such as hot components in the gas turbine engine. Previous TBC development and test has concentrated on ceramic overlays ranging from 0.25 mm to 0.38 mm in thickness. The high heat fluxes encountered in high pressure rocket engines dictate that thinner, 0.012 mm to 0.10 mm, coatings be used to avoid surface melting and mechanical failure of the coating.

The plasma sprayed TBC required in this application is a complex structure. No currently accepted model is known that relates coating performance to thickness and to controllable process variables. The use of very thin coatings, such as those under consideration, is particularly complicated. Four examples of this complexity are given below. These illustrate some of the difficulties that were anticipated in the practical and the analytical development program.

- 1. Structures are frequently two phase (e.g., monoclinic plus transformable tetragonal in the case of zirconia) and may vary from point to point.
- 2. Each successively deposited grain impacts a surface whose temperature may increase as coating thickness increases.
- 3. Defects (e.g.,pores) are of significant size in relation to coating thickness so that bulk or average coating properties may not be appropriate.
- 4. Individual grain size of the plasma spray powder approaches total coating thickness as does the normal surface roughness of plasma sprayed coatings.

2

PROCEDURES

2.1 LABORATORY MATERIALS AND PROCEDURES

The coating system selected for evaluation in this study was a copper substrate, NiCrAly bond coat and yttria stabilized zirconia ceramic thermal barrier. This bond coat/ceramic combination has been extensively investigated as a TBC for heat engines and shows great promise as a reliable coating.

The coating deposition performed in this study was accomplished using a Metco 7M plasma system. During the program, deposition parameters such as gas flow, power levels, standoff, etc. were varied. The methods used to define these variations, and tests used to establish the resulting effects on coating properties, are given in this section.

2.1.1 Powder Specification and Preparation

The powder materials used in this study were as follows:

```
Bond Coat = Amdry powder #961
18.3% Cr, 5.8% Al, 0.5% Y, bal. Ni
-170 + 325 mesh
```

Ceramic TBC = Cerac Zirconium Oxide-Yttria Stabilized powder $Zr0_2.8Y_20_3$ -325 mesh

The ceramic powder was determined to have the following size distributions:

```
+100 mesh - trace

-100 + 140 mesh - 4%

-140 + 200 mesh - 30%

-200 + 325 mesh - 61%

-325 mesh - 5%
```

Early tests indicated that narrowing the particle size range within that of the starting material had a negligible effect on coating structure. Thus, the as-received material was used in this study.

2.1.2 Plasma Variables

The following deposition parameters were varied during the test program. The effects of each variable on coating structure were evaluated by metallurgical examination.

Arc Current

During the program, current to the plasma gun was varied over a range of 300 to 1000 amperes. The current was measured at the control console using a 50 millivolt shunt and meter. Calibration tests indicated that, with manual correction for drift by the operator, an accuracy of ± 25 amperes could be maintained during a test run.

Arc Voltage

Arc voltage was also monitored at the control console and these values are reported to an accuracy of ± 2.5 volts. Voltage measurements at the gun input connections showed a voltage drop of 1 to 3 volts in the water cooled power cables. The voltage drop to the gun was dependent upon the current.

Plasma and Carrier Gas Flows

The flow of plasma on carrier Porter gas was controlled by Fisher purge meters. Pressure at the meters was maintained at 689 kPa. Flow was regulated by needle valves giving a reproducibility of $\pm 5\%$.

Powder Feed Rate

Powder feed control utilized a Model 1250 Plasmadyne hopper. This unit incorporates a metering disc with a digital speed readout to introduce powder into the carrier gas stream. Delivery rates were periodically verified by collecting and weighing the powder from the delivery tube over a period of thirty minutes. In addition, the initial weight of powder introduced into the cannister was recorded and at the completion of the test, the weight of powder remaining was established. The difference in these two weights was recorded. The latter method was subject to errors resulting from material retained in the powder feed system after emptying the cannister.

Deposition Rates

The rate of coating deposition was established by direct measurement. The plasma gun was traversed at a constant rate for a specified number of cycles. Thickness measurements were made after every five cycles to determine if deposit rates were constant. At completion of the test, metallurgical sections were made to verify coating thickness.

Nozzle Effects

The effect of changing the plasma nozzle (anode) were determined using the various configurations supplied for the Metco 7M system. No modifications were made in the base or configuration. Data supplied by the manufacturer were used as a guide in establishing basic operational characteristics.

2.1.3 Property Measurement

Density/Porosity

Five methods were used to determine coating density. In initial trials, test specimens were weighed before and after coating. The increased weight was attributed to the coating deposit. Coating thickness and area measurements were then made, the apparent volume calculated, and the apparent density (weight divided by volume) reported.

Porosity of the coating was also determined by immersing pre-weighed coated specimens in boiling water. The heat was removed and the water cooled to room temperature. The specimens were then removed from the water, excess water removed and the specimen rapidly weighed. The gain in weight was attributed to water entrapped in pores, the pore volume was then calculated using the known density of water.

The third and fourth methods used for determining coating density required the preparation of metallurgical coating sections. Test specimens were sectioned and photomicrographs were made, usually at 100% magnification. From these photomicrographs, porosity was established by photometrically comparing average reflectance of the coating section with the nonporous substrate and by direct measurement of the average pore area in a unit area of coating.

The fifth method used to determine density was by direct measurement. The area of a rectangular coated specimen was measured. The substrate and bond coat (if used) were carefully removed by chemical etching and the coating was weighed after rinsing and drying. The specimen was then mounted and polished for metallographic examination and coating thickness measurement. The volume of the coating was then calculated from this data and the density calculated using the previously determined coating weight.

Coating Thickness

Because of the extremely thin coatings being evaluated, thickness measurement required a high degree of precision. During the coating process, measurement of coating thickness was approximated using a micrometer. These measurements were limited to ± 0.013 mm by the roughness and irregularities in the coating. More precise measurements were made after coating the specimens using a metallurgical section and a microscope with a calibrated eyepiece.

Coating Strength

Coating strength was determined using a four point bend test (see Fig. 1). The analytical methods used to determine the coating stresses developed during this test are described in Appendix 1.

Copper specimens 0.81 mm thick x 13 mm wide x 76 mm long were coated on one side with a 0.03 mm NiCrAlY bond coat. A yttria stabilized TBC was then applied over this bond coat at selected thicknesses ranging from 0.02 mm to 0.15 mm. The specimen was then placed on two knife edge supports spaced a distance of 51 mm apart. For tensile loading of the coating the coated surface was placed on the supports and for compressive coating loads the coated surface was placed opposite from the supports. The specimen was then loaded symmetrically by knife edges spaced 25 mm apart at a rate of 0.08 mm/second until coating failure occurred as evidenced by an abrupt decrease in the load required to maintain a constant rate of strain. Deflection was initially measured using a dial gauge mounted directly below the mid-point of the specimen. Subsequent tests used crosshead travel during loading to monitor specimen deflection.

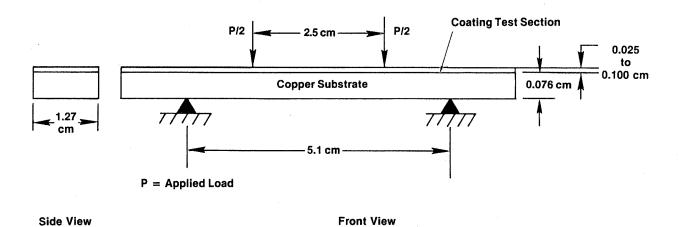


Figure 1. Schematic of Four Point Flexure Test

Thermal Diffusivity and Conductivity

Thermal conductivity of the coatings were measured at Purdue Laboratories using a laser heat pulse technique. A detailed discussion of this technique is given by Taylor (Ref. 1).

For this measurement, copper specimens $13 \text{ mm} \times 13 \text{ mm} \times 1.6 \text{ mm}$ thick were coated on one side with the TBC to be tested. Coating thickness was a nominal 0.18 mm. A thermocouple was attached to the uncoated surface of the specimen, the assembly furnace heated to the desired test temperature and a laser heat pulse applied to the coating. Temperature rise of the back face as a function of time was measured. This, combined with coating and substrate thicknesses, densities and specific heats enabled the calculation of diffusivity and conductivity using the methods described by Taylor in the previously cited reference.

2.1.4 Calorimeter Testing

A water cooled calorimeter was used to measure average heat flux from the plasma torch, heat flux during coating deposition and the reaction of the coatings to high heat inputs. This calorimeter, shown in Figure 2, consists of two water cooled copper components. The first is the central test area and the second is the outer guard ring used to shield areas other than the test surface from external influences. In operation, a constant metered flow of water is supplied to the test area, and water inlet and outlet temperatures are monitored. The calorimeter is inserted into the plasma flame and the heat absorbed by the cooling water is presumed to be the total heat input over the test area.

The calorimeter data is then converted to heat flux in the following manner:

$$Q = \frac{(0.279) (W) (\Delta T)}{\pi d^2}$$

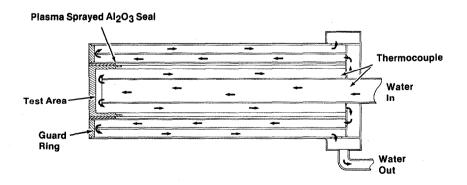


Figure 2. Calorimeter

where:

```
Q = heat flux (watts/mm<sup>2</sup>)

\Delta T = Tinlet - Toutlet (°C)

W = water flow (grams/min.)

d = test diameter (mm)
```

2.1.5 Residual Stress and Stress Free Temperature

Residual stress in the coating was evaluated using the techniques described by Andrews (Ref. 2). The model is shown in Figure 3 and the symbol definitions are given below:

Subscript (1) ceramic
Subscript (2) copper

\alpha = coefficient of thermal expansion
E = modulus of elasticity
I = moment of inertia
h = total composite thickness
a_1 = ceramic thickness
a_2 = copper thickness
L = active length
r = radius of curvature = L2/2
\delta = tangential deflection
S_1 = stress in ceramic

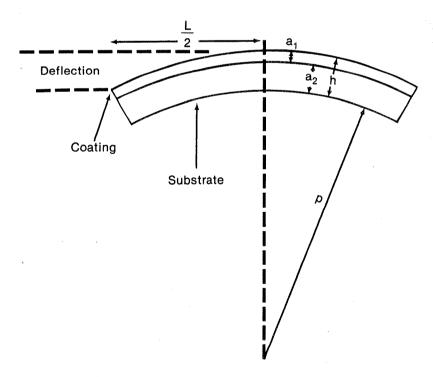


Figure 3. Residual Stress Model

$$\begin{array}{rcl} b & = & \text{specimen width} \\ \Delta T & = & T_{SF} - T_{ambient} \\ T_{SF} & = & \text{stress-free temperature} \end{array}$$

This approach was initially developed for calculation of stresses in porcelain enamels and fits well with the requirements of thin TBC's. The methodology is described by the following equation:

$$S_1 = \frac{-2\delta}{L^2} \left[\frac{2}{ha_1} \left(E_1 I_1 + E_2 I_2 \right) + \frac{a_1 E_1}{2} \right]$$
 (1)

Since deflection (δ) can be measured directly, equation (1) is easily applied.

For most calculations E_1I_1 and a_1E_1 are small and can be omitted from the calculation, reducing equation (1) to:

$$S_1 = \frac{-4\delta E_2 I_2}{L^2 ha_1}$$
 (2)

where

$$I = \frac{b a^3}{12} \tag{3}$$

The stress-free temperature of the system is calculated from the bi-metallic relationship:

$$\delta = K \frac{(\alpha_2 - \alpha_1) (\Delta T) L^2}{h}$$
 (4)

where

$$K = \frac{3(1 + m)^2}{3(1 + m)^2 + (1 + m) (m^2 + \frac{1}{mn})}$$
 (5)

and

$$m = \alpha_1/\alpha_2 \tag{6}$$

$$n = E_1/E_2 \tag{7}$$

Hence:

$$\Delta T = \frac{h}{(\alpha_2 - \alpha_1)L^2K}$$
 (8)

and

$$T_{SF} = \Delta T + 20^{\circ}C \tag{9}$$

For the purposes of these calculations the following constants were used:

$$E_1 = 45 \times 10^6 \text{ kPa}$$

 $E_2 = 117 \times 10^6 \text{ kPa}$
 $\alpha_1 = 7.15 \times 10^{-6}/^{\circ}\text{C}$
 $\alpha_2 = 17.6 \times 10^{-6}/^{\circ}\text{C}$

The following procedures were used to obtain the deflection values (δ) used in these calculations. Copper test strips, 0.8 mm x 13 mm x 76 mm, were bond coated with 0.03 mm of NiCrAlY on one side. The deflection, or chord height, if any, was determined over the 50 mm midspan (L). The specimen was then coated with the zirconia TBC and the measurement repeated. Any change in curvature was recorded and the resulting value used to calculate the residual stress in the coating.

Stress free temperatures were also calculated from this data. The results were then confirmed by furnace heating the specimen to the calculated temperature and determining if it returned to its original configuration.

Similar tests were performed using copper rings 68 mm in diameter rolled and butt welded from 13 mm wide, 0.8 mm strips. The rings were vacuum annealed at 760°C after bond coating to relieve initial forming and coating stresses. After coating and segmenting, the change in diameter of the ring, due to the coating process, was established and used in Equation (1) for an alternate means of calculating residual stress.

2.1.6 Substrate Temperature

During trial coating application, the substrate temperature was monitored using an Inconel sheathed chromel-alumel thermocouple, 0.76 mm in diameter. The thermocouple was installed in the copper substrate by first drilling a 0.77 mm hole through the specimen and then inserting the thermocouple. The edges of the hole were then staked to retain the thermocouple with its tip flush with the surface to be coated. Then a NiCrAly bond coat was applied to bond the tip of the thermocouple to the substrate. Temperature recording was done with a conventional strip chart recorder.

2.2 ANALYTICAL PROCEDURES

2.2.1 Coating System Performance Analysis

Finite element analyses, using the ANSYS code, were used to evaluate coating system performance under simulated operating conditions. The coating system model used for these analyses (shown in Fig. 4) simulates the coated thrust

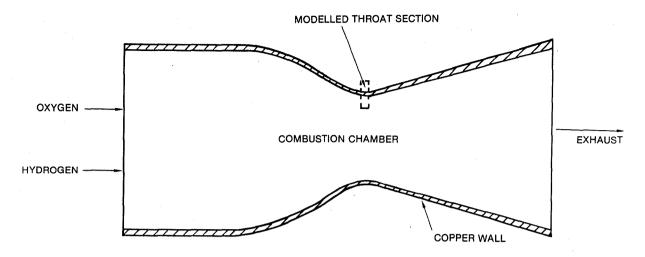


Figure 4a. Rocket Thrust Chamber

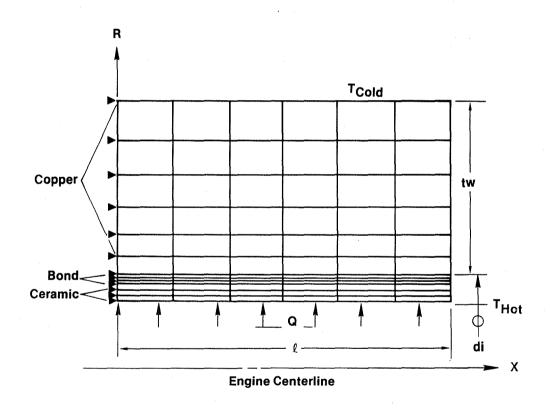


Figure 4b. Thrust Chamber Model

chamber wall of a rocket engine. This two-dimensional, axisymmetric model represents a cross-section of the wall at the throat of the thrust chamber.

The elements used in this finite element model were two-dimensional, isoparametric, axisymmetric ring elements. Each element was defined by four nodal

points and each node was defined by a radius and an axial location. For thermal analyses, each node had a single degree of freedom: temperature. For structural analyses, each node had two degrees of freedom: translations in the radial and axial directions.

The finite element geometry for the thermal analyses was identical to that for the structural analyses. This allowed the temperature distributions which were derived from the thermal analyses to be directly translated to the structural model.

The basic geometry of the thrust chamber model consists of a 66.04 mm internal diameter (di) copper cylinder with a 0.889 mm wall (tw). Attached to the inner surface of this cylinder is a 0.038 mm thick (tb) NiCrAlY bond coat, and a variable thickness (tc) $\text{Zr0}_28\text{Y}_20_3$ ceramic coating. Ceramic coating thicknesses from 0.0127 mm to 0.203 mm were investigated. The axial length (ℓ) of the model was 0.229 mm, although a 50.8 mm length was used in some of the preliminary analyses.

The finite element model was divided into nine equal elements in the axial direction, to accommodate axial variations in boundary conditions. In the radial direction, the ceramic and bond coats were divided into three element layers each, and the copper substrate was divided into six. This element configuration resulted in element aspect ratios (axial length to radial thickness ratios) from 0.375 to 6 in the ceramic coating, 2 in the bond coat and 0.17 in the copper substrate. Use of the long (50 mm) preliminary model resulted in element aspect ratios from 83 to 667 in the ceramic and 444 in the bond coat. It is generally desirable to maintain an aspect ratio between 1 and 4, to ensure numerical accuracy. The effect of aspect ratio on the accuracy of these analyses was evaluated by comparing results using the long and short finite element models.

The three coating system materials were assumed to be homogeneous and isotropic. Properties of all three materials were initially obtained from the literature. These properties are listed in Table 1. In the final analyses, measured material properties were substituted for literature values where applicable. The temperature dependency of these material properties created a non-linear problem. This required iterative analyses, with the solutions defined using a specified measure of convergence.

A variety of thermal boundary conditions were selected for these coating systems analyses. These conditions were intended to simulate the range of anticipated operating conditions in a typical high pressure rocket engine thrust chamber and to define an operating envelope based on coating material limitations. Thermal boundary conditions were established in two ways: (1) a hot surface temperature and a cold surface temperature were defined or (2) a heat flux and a cold surface temperature were defined.

The outer surface of a typical rocket thrust chamber is cooled with liquid hydrogen, which has a boiling point of -240°C at ambient pressure. Test data indicate that the actual cold surface temperature of a rocket chamber during operation is between 0 and 200°C. For these analyses, three cold surface temperatures were evaluated: -240, 0, and 200°C. Heat fluxes in the range

 $\label{eq:Table 1} \mbox{Material Properties From the Literature (Refs. 10-16)}$

Material	Thermal Conductivity (W/mK)	Elastic Modulus (GPa)	Specific Heat (J/kg.K)	Thermal Expansion Coefficient (x 10 ⁻⁶ /C)	Density (g/cc)	Poisson's Ratio
ZrO ₂ 8Y ₂ O ₃ Ceramic Coating	2.2 x 10 ⁻⁴ T + 1.09 (400 <t<2200°k)< td=""><td>45.06 - 0.0248T (90°C<t<1200°c)< td=""><td>317 + 0.813T -9.55 x 10⁻⁴T² + 6.30 x 10⁻⁷T³ (200°K<t<1500°k)< td=""><td>7.1844+0.003888T (90°C<t<1200°c)< td=""><td></td><td>0.23</td></t<1200°c)<></td></t<1500°k)<></td></t<1200°c)<></td></t<2200°k)<>	45.06 - 0.0248T (90°C <t<1200°c)< td=""><td>317 + 0.813T -9.55 x 10⁻⁴T² + 6.30 x 10⁻⁷T³ (200°K<t<1500°k)< td=""><td>7.1844+0.003888T (90°C<t<1200°c)< td=""><td></td><td>0.23</td></t<1200°c)<></td></t<1500°k)<></td></t<1200°c)<>	317 + 0.813T -9.55 x 10 ⁻⁴ T ² + 6.30 x 10 ⁻⁷ T ³ (200°K <t<1500°k)< td=""><td>7.1844+0.003888T (90°C<t<1200°c)< td=""><td></td><td>0.23</td></t<1200°c)<></td></t<1500°k)<>	7.1844+0.003888T (90°C <t<1200°c)< td=""><td></td><td>0.23</td></t<1200°c)<>		0.23
Solid Zirconia (Stabilized ZrO ₂)	2.708 - 5 x 10 ⁻⁴ T (70°C < T < 1100°C)	205	400 to 640	8.6 to 15.2	5.63 to 6.27	0.23 to 0.35
NiCrAlY Bond Coat	8.3 x 10 ⁻³ T + 6.7 (400°K <t<1400°k)< td=""><td>125.6-0.0149T (90°C<t<550°c)< td=""><td>0.476 + 0.00086T -0.0000015T² (90°C<t<1100°c)< td=""><td>13.3182-0.0121T + 5.8729E-5T² (90°C<t<1100°c)< td=""><td>6.98</td><td>0.23</td></t<1100°c)<></td></t<1100°c)<></td></t<550°c)<></td></t<1400°k)<>	125.6-0.0149T (90°C <t<550°c)< td=""><td>0.476 + 0.00086T -0.0000015T² (90°C<t<1100°c)< td=""><td>13.3182-0.0121T + 5.8729E-5T² (90°C<t<1100°c)< td=""><td>6.98</td><td>0.23</td></t<1100°c)<></td></t<1100°c)<></td></t<550°c)<>	0.476 + 0.00086T -0.0000015T ² (90°C <t<1100°c)< td=""><td>13.3182-0.0121T + 5.8729E-5T² (90°C<t<1100°c)< td=""><td>6.98</td><td>0.23</td></t<1100°c)<></td></t<1100°c)<>	13.3182-0.0121T + 5.8729E-5T ² (90°C <t<1100°c)< td=""><td>6.98</td><td>0.23</td></t<1100°c)<>	6.98	0.23
Oxygen Free Copper Substrate	391	117	0.39	5.4	8.94	0.350

typically encountered in rocket thrust chambers, 50 to 165 W/mm 2 , were also used for these analyses. Typically values of 49, 82 and 164 W/mm 2 were used. Ceramic surface temperatures were evaluated in the range of 1000 to 2600°C ($\rm Zr0_28Y_20_3$ melting point). A maximum desirable ceramic operating temperature of $1650^{\circ}\rm C$ was established based on past experience, and this value was used for most of the analyses in which a ceramic surface temperature was specified.

One structural boundary condition was imposed for these analyses: one radial edge of the coating system model was restrained from movement in the axial direction. This condition was held for all analyses. No external loads, pressures, or displacements were imposed.

The thermal and structural boundary conditions are indicated graphically in Figure 4. Examples of the ANSYS finite element input and output are included in Appendix 2. The output from the thermal analyses consisted of heat flow rates and temperature distributions. The output from the structural analyses consisted of displacements, stresses, and reaction forces.

2.2.2 Coating Structure Analysis

The effect of coating microstructure on the thermal conductivity of a coating was evaluated using the ANSYS finite element code. A two-dimensional, planar model of a coating system was used for these analyses. This model, shown in Figure 5, used a biaxial plane thermal element. Each element was defined by four nodes having a single degree of freedom, temperature, at each node.

The model shown in Figure 5 represents a simplified porous coating structure. This geometry was selected to facilitate the finite element modeling. The model was essentially composed of "blocks" of coating material. Pores were simulated by eliminating some of these blocks, in specified patterns.

The model used in these analyses simulates the basic coating system being considered by NASA for rocket thrust chambers: a 0.889 mm thick copper substrate with a 0.038 mm NiCrAlY bond coat and a variable thickness $\rm Zr0_28Y0_3$ ceramic coat. The material properties that were used are presented in Table 1. The value of thermal conductivity for the ceramic in Table 1 was used for the solid ceramic elements in the finite element model.

Boundary conditions for this model were established by specifying the hot ceramic surface temperature and the cold copper surface temperature. To maintain an accurate temperature gradient along the edges of the coating, the nodes along each edge were coupled to similar nodes within the coating.

The effect of radiation across the individual pores was investigated using this model. Radiation links were specified between the upper and lower surfaces of the pores. Radiative heat transfer was calculated from the standard radiation function defined as follows:

$$q = \sigma \Sigma F A (T_1^4 - T_2^4)$$

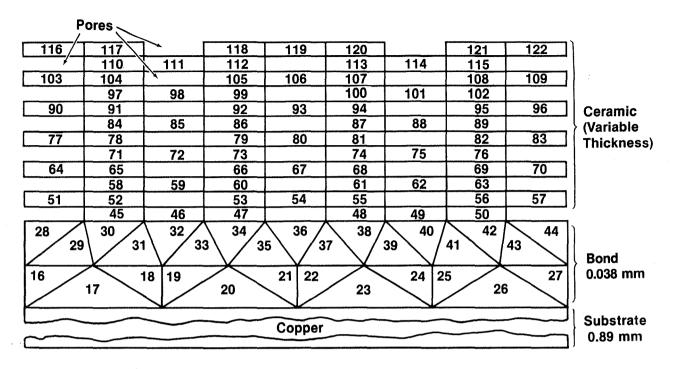


Figure 5. Finite Element Model of a Porous Coating

where:

Stefan-Boltzmann constant

 T_1 = temperature of hot surface T_2 = temperature of cold surface

Σ emissivity

F geometric form factor

Α area

heat flow rate

The effect of coating microstructure on the thermal conductivity of ceramic coatings was investigated using this finite element model. Various coating structures were evaluated, having different pore sizes, pore shapes, pore densities, etc. Each coating structure was characterized by three geometric parameters:

- $P_{\mbox{\scriptsize t}}$ total porosity; the volume of pores divided by the total 1. coating volume (% of total volume)
- D_r dimension ratio; the pore length (axial) divided by the pore 2. thickness (radial)
- P_{i} pore size; total porosity divided by the number of pores 3. (% of total volume)

The goal of these analyses was to establish a relationship between the three coating variables (D_r, P_t, P_i) and effective thermal conductivity.

The approach was to subject each coating structure to a fixed temperature differential and calculate, using finite element analyses, the heat flow through the coating. The effective thermal conductivity of each coating structure was then calculated from

$$k = 0 t/A \Delta T$$

where

k = effective thermal conductivity of the coating

Q = heat flow through the coating

A = overall area of coating model perpendicular to heat flow

 ΔT = temperature drop across coating

t = coating thickness

The effective conductivity of each coating structure was normalized to the maximum possible thermal conductivity. This maximum value was established by analyzing a coating with no porosity. Thus, each conductivity is expressed as a percent of maximum.

This analytical model of the coating structure did not consider thermal contact resistance between layers of ceramic or between different materials. It was anticipated that the effect of thermal contact resistance, and any other unaccounted for phenomena, would be reflected in an empirical adjustment of the analytical model. Conflicting reports as to the significance of thermal contact resistance in TBC's have been observed in the literature (Ref. 3, 4).

2.2.3 Failure Strain Analyses

The four-point flexure test was initially evaluated using coating strength data from the literature (Ref. 5). This reference quoted a compressive failure strain of 0.78% in a Mg0-Zr02 coating, using a similar four-point flexure test. It was calculated that hardened copper could accommodate this strain level without plastic deformation. An elastic stress-strain analysis was thus derived, for a composite beam, to evaluate the failure strains in the coated specimens.

Preliminary testing with the four point flexure test for measuring coating strengths revealed that plastic deformation of the copper substrate occurred before coating failure. The initial stress-strain analysis, derived for an elastic composite beam specimen, was then replaced by a more sophisticated analysis which accounted for plasticity in the copper substrate. This analysis is described in Appendix 1 with the computer program created to perform the numerical solution.

2.3 COATING OF SUBSCALE THRUST CHAMBER

Initial development work on this program was accomplished using flat specimens and a standard plasma spray gun configuration. The test chambers were 66 mm diameter cylinders and were not accessible to the standard gun configuration. A 61 cm small bore extension gun was therefore used to coat these chambers. The two gun configurations are shown in Figure 6.

In the conventional configuration, the electrode, plasma gas flow and coating deposition were along a single axis normal to the coated surface. Gun stand-off was varied over a range of 50-100 mm to aid in process control. The small bore coating equipment deflects the plasma gas stream at an angle of 45° to the gun axis, and to the coated surface. Gun standoff was limited by the cylinder ID. Further coating development tests were required to produce equivalent coatings with changed equipment.

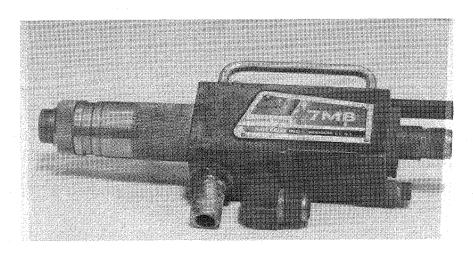
The evaluation procedure consisted of first establishing three coating structures using conventional test specimens as described in Section 2.1.2. Metallurgical and material property evaluations were used to characterize each coating. Test rings 66 mm in diameter, formed from 0.81 mm thick copper strips, were then coated using the angle extension. Using the previously determined power levels, the gas flows and powder feed rates were varied until equivalent structures were obtained with the internal diameter equipment. In addition, the ring was periodically reversed relative to the gun entry end to compensate for the 45° impingement angle and offset the shadowing effects of previously deposited coating particles.

To verify coating quality on the test cylinder, extensions of the same diameter were attached to each cylinder and coated simultaneously using the settings established by the previous ring tests. After the coating operation was completed, these extensions were removed, sectioned and evaluated metall-urgically.

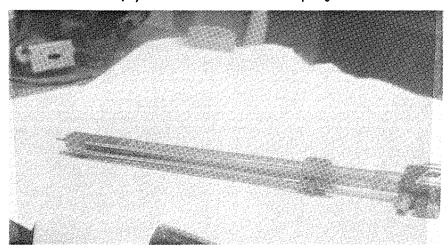
The use of thermocouples to measure substrate temperatures during deposit was not feasible due to the double wall construction of the test chamber. Temperature recorders* in the form of adhesive strips which indicated the maximum temperature by a permanent color change were located in selected locations on the thrust chambers. These provided information on the maximum temperature developed during the coating operation.

The fixture used to coat the chamber is shown in Figure 7. The thrust chamber was located on a variable speed, rotating head. For control of substrate temperature, a controlled cooling air flow was introduced through the fuel line fittings on one end. Exhaust air temperature was monitored to further insure part-to-part repeatability.

^{*} Manufactured by Tedatemp Corporation, Fullerton, CA



(A) Metco 7MB Direct Spray Gun



) Metco 7MBT-12 45 Degree Angle Spray Gun Figure 6. Two-Gun Configurations

The procedure consisted of mounting the thrust chamber as shown and abrasive blasting the internal diameter while rotating the assembly. The bond coat and ceramic overlay were then plasma sprayed using the angle extension. The operation was interupted during application of both the bond and ceramic coats after a thickness of 0.01 mm had been applied. The chamber was removed from the fixture and remounted on the opposite end to compensate for the directionality of the spray and provide a more uniform coating structure. This reversal was repeated periodically throughout the coating cycle after each 0.01 mm, until full coating thickness was achieved.

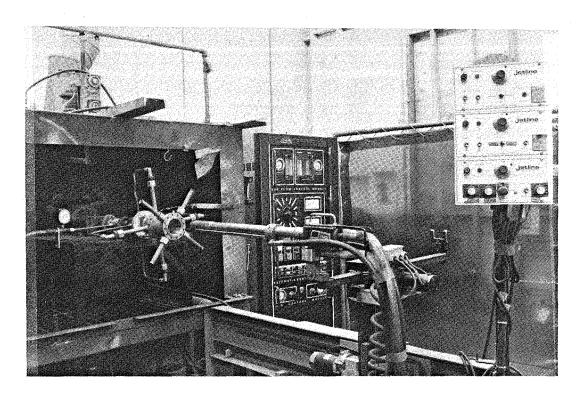


Figure 7. Coating Fixture

3

RESULTS

3.1 PLASMA DEPOSITION VARIABLES

Results of tests to determine the effects of plasma deposition parameters on TBC's are reported in Sections 3.1 and 3.2. Section 3.1 reports the general effects of varying process parameters on the resulting coating. These tests were used to select three sets of deposition parameters for material property evaluation. The results of the property measurements for the three coatings are given in Section 3.2.

3.1.1 Effect of Plasma Gas Flow on Coating Characteristics

For these tests the basic parameters presented in Table 2 were used with the Metco 7M system for deposition of the NiCrAly bond coat.

The first variable selected was plasma gas flow. The results of these tests are presented in Table 3 and Figures 8 through 16. At this time no attempt was made to optimize the coating but only to determine the effect of an individual parameter (gas flow) on coating structure.

Based on thickness measurements alone, coating deposit rate was established as a function of plasma gas flow. The deposit rate decreased steadily with increasing gas flow. At a gas flow of $2.69~\text{m}^3/\text{hr}$ no measurable deposit could be obtained.

The photomicrographs shown in Figures 8 through 16, illustrate the effect of gas flow on coating structure. At the lowest gas flow $(1.70~\text{m}^3/\text{hr})$ coating buildup was irregular with large voids. The asperities visible in Figure 8 show the peaks that give rise to the apparent rapid coating buildup. As flow was increased, the coating tended towards a denser structure with some large voids. At a gas flow of $2.12~\text{m}^3/\text{hr}$ the spherical unmilled particles observed at lower gas flow tended to disappear although some porous regions existed in the coating as shown in Figure 11. The most uniform coating obtained with these parameters was developed at a gas flow of $2.26~\text{m}^3/\text{hr}$. As the gas flow was further increased, the coating tended to deteriorate with only the larger particles having deposited. At the maximum flow of $2.69~\text{m}^3/\text{hr}$ only an occasional deposited particle was observed and the substrate was significantly oxidized.

Similar tests were performed to determine the effect of plasma gas flow on the ceramic coating structure. For these test the parameters in Table 4 were used as a starting point.

Table 2
Basic NiCrAly Test Parameters

Current 500 amperes Voltage 60 volts Argon Flowrate Variable Hydrogen Flowrate 0.28 m³/hr Standoff Distance 3.1 cm Powder Port No. #2 Carrier Argon Flowrate 0.28 m³/hr Powder Feedrate 5.7 gm/min. Cooling Air Jet Pressure 550 kPa Nozzle Model No. 713 Traverse Rate 2.5 cm/sec
Substrate Material 0.81 mm copper

Table 3

Effect of Plasma Gas Flow on Deposit Rate of NiCrAlY

Gas/Flow (m ³ /hr)	Deposit Rate (mm/traverse)				
1.70 1.84 1.98 2.12 2.26 2.40 2.55 2.69	0.016 0.012 0.0074 0.0064 0.0031 0.0021 0.0010				
*Too small to measure.					

Monitoring of the substrate temperature during coating deposition was accomplished using a 1.6 mm Inconel sheathed thermocouple inserted into a hole drilled through the specimen. It was located so that its tip was flush with the substrate surface.

A photomicrograph of the baseline ceramic coating is shown in Figure 17. The average thickness of the ceramic is 0.09~mm. The individual coating particles can be distinguished at $1000 \times 1000 \times 1000$

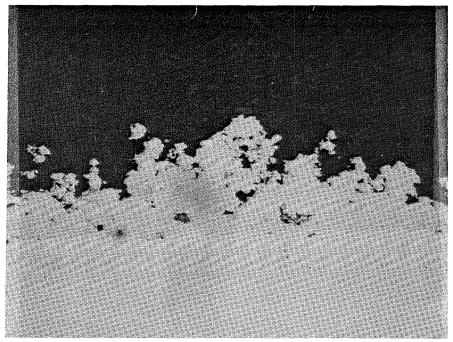


Figure 8. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 1.70 $\rm m^3/Hour$

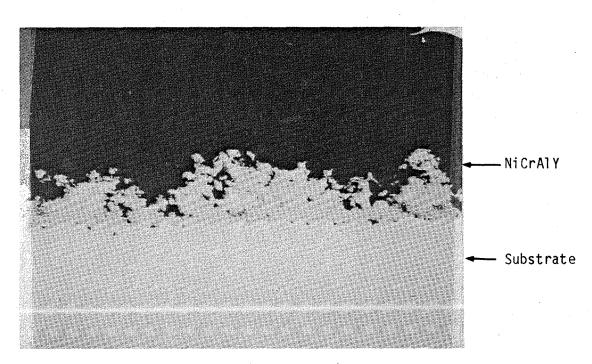


Figure 9. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 1.84 $\rm m^3/Hour$

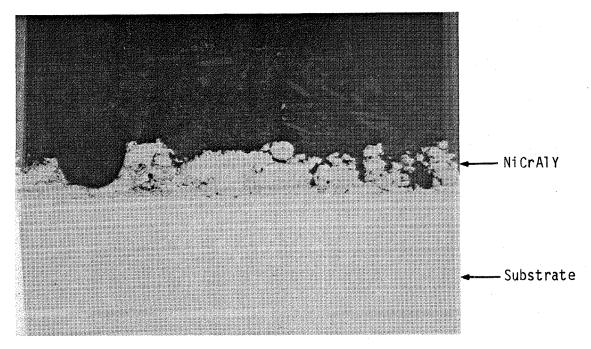


Figure 10. NiCrAlY Bond Coat Applied for 12 Cycles at Gas Flow of 1.98 $\rm m^3/Hour$

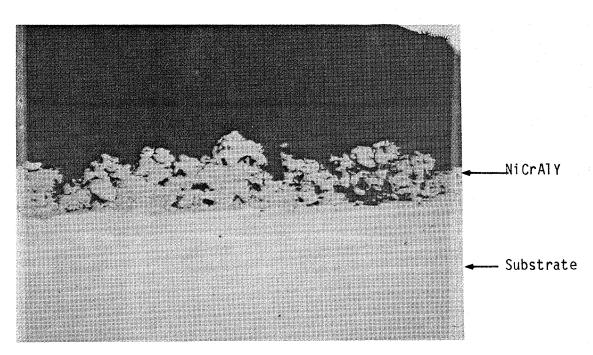


Figure 11. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.12 $\mathrm{m}^3/\mathrm{Hour}$

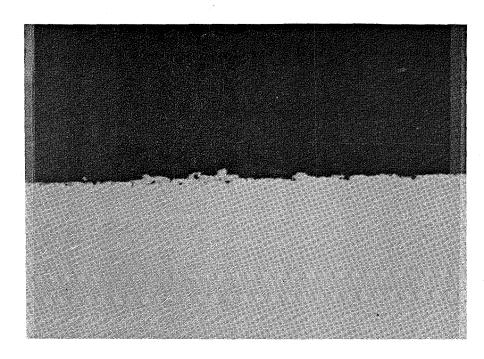


Figure 12. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.15 $\rm m^3/Hour$

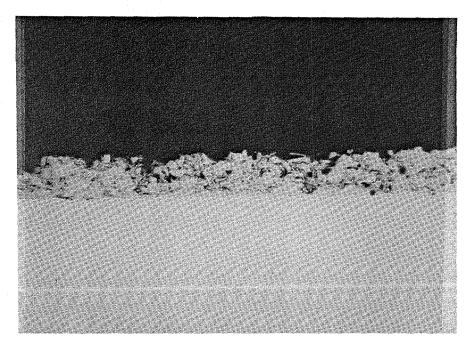


Figure 13. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.26 $\rm m^3/Hour$

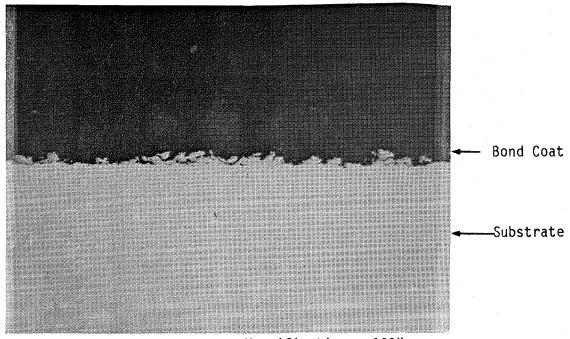


Figure 14. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.40 $\rm m^3/Hour$

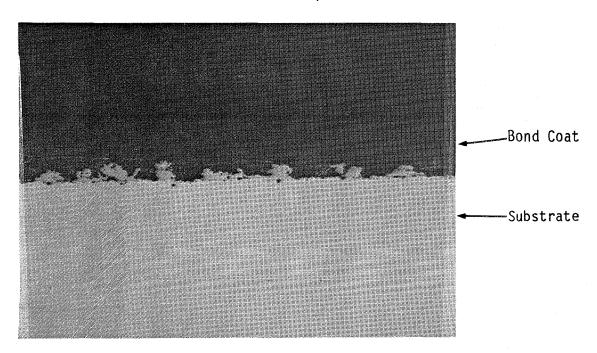


Figure 15. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.40 $\rm m^3/Hour$

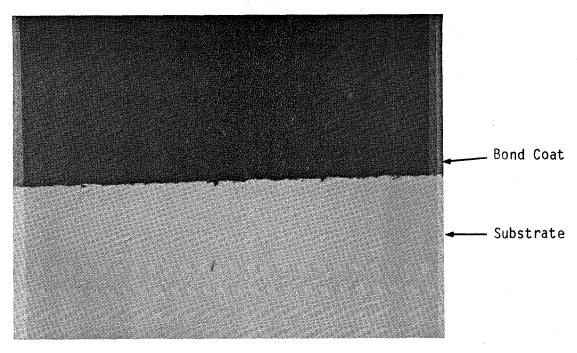
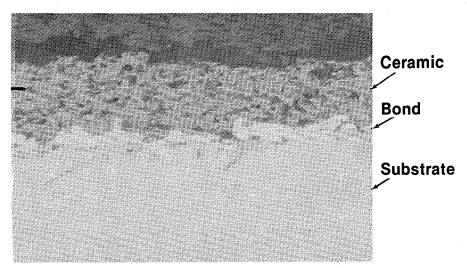


Figure 16. NiCrAlY Bond Coat Applied for 12 Cycles at a Gas Flow of 2.69 $\rm m^3/Hour$

Table 4
Baseline Ceramic Coating Spray Parameters

Current	400 amps
Voltage	50 yolts
Argon Flowrate	$1 \text{ m}^3/\text{hr}$
Hydrogen Flowrate	0.23 m ³ /hr
Standoff Distance	10.2 cm
Cooling Air Pressure	551 kPa
Deposit Rate (avg)	0.008 mm
Traverse Rate	2.5 cm
Powder Feedrate	3.0 gms/min.
Substrate Temperature	97-119°C
Coating Thickness	0.10 mm

Some voids and numerous microcracks are also visible. Increasing the plasma gas flow to 1.15 $\rm m^3/hr$ and then to 1.43 $\rm m^3/hr$ did not produce a visible change in microstructure. However, it was observed that this gas flow rate increase decreased the rate of deposition by 42 percent. The substrate temperature also increased with increasing gas flow, by 80°C in the first case and by 13°C in the second.



Magnification: 200X

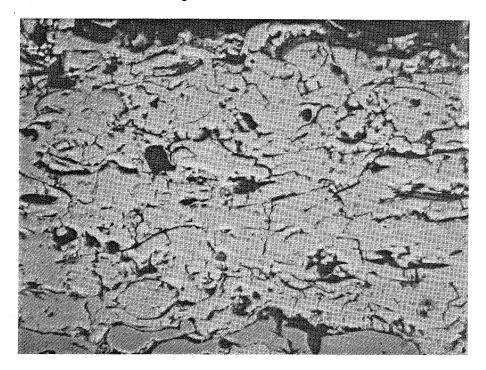


Figure 17. Baseline Thermal Barrier Coating (Mount No. 5263)

3.1.2 Standoff Distance Effects

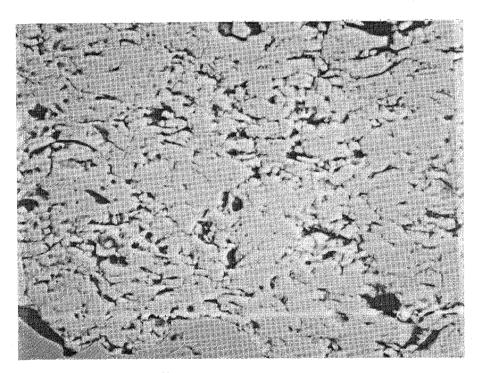
The effects of standoff distance (distance between gun and substrate) are tabulated below. This data was generated using the baseline parameters given in Table 4.

Standoff	Substrate Temperature	Deposit Rate (per cycle)	Effect on Coating Structure
10.2 cm*	97-119°C	0.008 mm	Figure 17
9.0 cm	108-128°C	0.008 mm	Slight porosity decrease
7.6 cm	144-167°C	0.008 mm	Larger interlaminar structure
11.4 cm	129-153°C	0.003 mm	Large particles entrapped

*Baseline coating

Decreasing the standoff distance to 9.0 cm increased the coating density slightly as shown in Figure 18. The substrate temperature also increased to a maximum of 132° C. Further reducing the standoff to 7.6 cm increased the size of the agglomerated particles and caused wider laminar separations as shown in Figure 19. Substrate temperature also increased to a maximum of 167° C during coating.

Increasing gun standoff to 11.4 cm produced a coating with poor integrity as shown in Figure 20. During normal plasma spraying, with the selected spray parameters, the cooling jet was adjusted to impinge at the point where the plasma hit the substrate. The jet served to cool the substrate and to remove unmelted nonadherent particles.



Magnification: 1000X

Figure 18. Thermal Barrier Coating With Gun Stand-off Distance of 9.0 cm

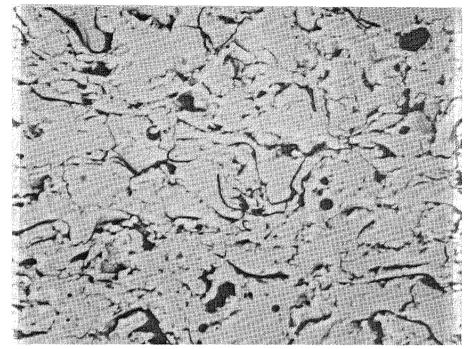


Figure 19. Thermal Barrier Coating With Gun Stand-off Distance of 7.6 cm

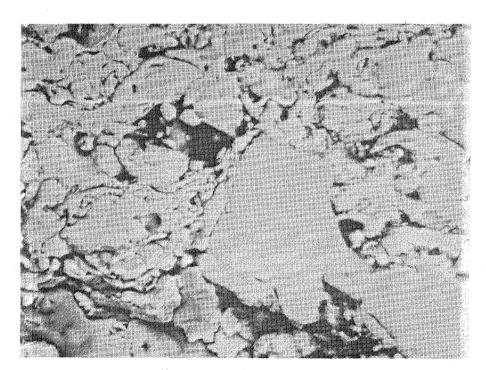


Figure 20. Thermal Barrier Coating With Gun Stand-off Distance of 11.4 cm $\,$

At 11.4 cm the effectiveness of the jet was greatly diminished. Large unmelted particles were entrapped in the coating and the substrate temperature increased to a maximum of 153°C, approximately midway between the temperature observed at 7.6 cm and 9.0 cm. Also, particle temperatures decreased to the point where interparticle bonding was decreased. Deposit efficiency also dropped to 40 percent of that recorded at the baseline standoff.

Attempts to spray at standoffs greater than 11.4 cm were unsuccessful. No measurable deposit was obtained.

3.1.3 Arc Voltage Effects on Coating Structures

In the plasma spray system, arc voltage (a dependent variable) is a function of the plasma gas composition. As the amount of hydrogen in the argon/hydrogen plasma gas mixture is raised, gas enthalpy, heat transfer and arc voltage all increase. For practical reasons, this effect is generally controlled by adding sufficient hydrogen to produce the desired voltage. Data is therefore reported on the basis of voltage.

The baseline parameters in this study are those reported in Table 2 except that hydrogen flow was adjusted to vary the voltage. Results are summarized below:

Voltage	Substrate Temperature (°C)	Deposit Rate (mm per Cycle)	Effect on Coating Structure
60*	97-119	0.008	Figure 17
66	108-130	0.008	None definable
72	149-171	0.007	Slight densification
78	143-152	0.006	Laminar separation

^{*}Baseline coating

Increasing the voltage from the baseline of 60 volts to 66 volts had no visible effect on coating quality. A substrate temperature increase of 11°C was observed. The coating is shown in Figure 21. When the voltage was raised to 72 volts (Fig. 22) interlaminar bonding appeared to improve and the size of the large voids was reduced. Also, less evidence of ceramic bond coat voids was evidenced. The substrate temperature also increase by 52°C .

Further increasing the voltage to 78 volts tended to produce laminar separation in the coating. The deposit rate decreased by 29 percent and substrate temperatures were 5° C lower than at 72 volts, due to the lower density of the coating. Voltage could not be increased beyond this point due to equipment limitations.

The effect of voltage on the bond coat was established using a 61 cm angle extension gun. This unit was selected for these tests since it was capable of

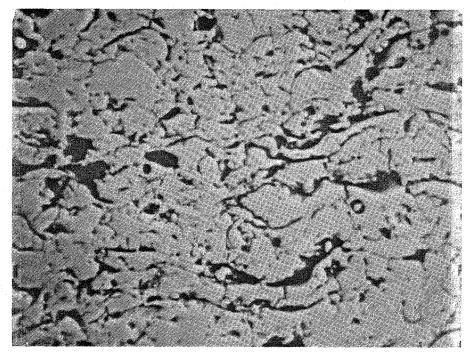


Figure 21. Thermal Barrier Coating With Plasma Arc Voltage of 66 Volts

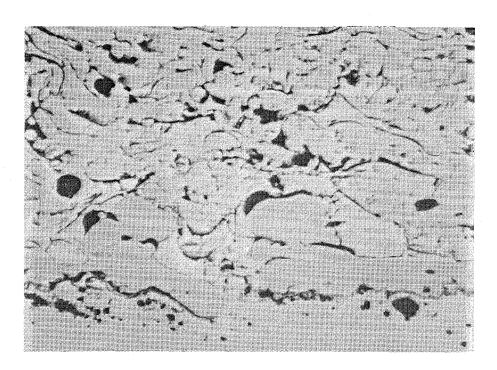


Figure 22. Thermal Barrier Coating With Plasma Arc Voltage of 72 Volts

coating the 6.6 cm base of a test thrust chamber. The parameters in Table 5 were used as a baseline for these tests.

Table 5
Baseline, Angle Extension Spray Parameters

Current	500 amperes
Voltage	45 volts
Argon Flowrate	1.7 m ³ /hr
Hydrogen Flowrate	0.8 m ³ /hr
Standoff Distance	3.2 cm
Powder Port Type	#2
Carrier Gas Flowrate	0.28 m ³ /hr
Powder Feedrate	0.098 gm/sec
Anode Model	#713 °
Traverse Rate	2.5 cm/sec
Deposit Rate	0.0025 mm/traverse

Table 6
Effect of Voltage on Bond Coat Structure

Voltage	Hydrogen Flowrate (m ³ /hr)	Deposit Rate (mm/cycle)	Deposit Width (mm)
45	0.085	0.0025	not definable 3.2 4.8 6.4 7.9
50	0.20	0.0051	
55	0.33	0.010	
60	0.51	0.020	
65	0.65	0.028	
70	0.76	0.033	
75	0.82	0.038	

Table 6 summarizes the measured results of these tests. The relationship between arc voltage and hydrogen flow proved to be linear. The width over which the deposit occurred also increased linearly with increasing hydrogen. The width measurement was made directly using a machinists scale. Thus, the accuracy of this measurement is limited. The increase over a 5 volt range was less than the measured accuracy (+0.5 mm) making this relationship less definitive. The deposit rate increased rapidly over the 55 to 65 volt range indicating that this range is especially sensitive to voltage variations.

The effect of voltage variations on bond coat deposition is shown in Figure 23 to Figure 29. In each case the plasma spray gun was traversed across the specimen twelve times.

In Figure 23 the arc voltage was 45 volts. There is no visual evidence of bond coat deposition. Oxidation of the substrate is apparent. At 50 volts, coating deposition has begun. In Figure 24 small local areas evidence individual particle deposition. The coating is sporadic and again the copper substrate is oxidized. The effect of increasing the voltage to 55 volts is evident in Figure 25, where significant bond coat deposition has occurred. The oxidation of the substrate observed in the two previous specimens is absent and the bond-substrate interface is relatively clean. (In viewing these photomicrographs the main deposit is in the central third. Material at each end is overspray.)

At 60 volts the coating thickness is nearly twice that of the coating deposited at 55 volts. The substrate is just beginning to evidence overheating at the edges. The effect of the 45 degree plasma jet impingement angle is also evident. The coating buildup slopes to the left with elongated voids that are created by the masking effects of previously deposited particles. With the voltage increased to 65 volts, the coating substrate bond shows evidence of significant oxidation. Wide variations in coating density were also observed, ranging from large porous areas to high density regions.

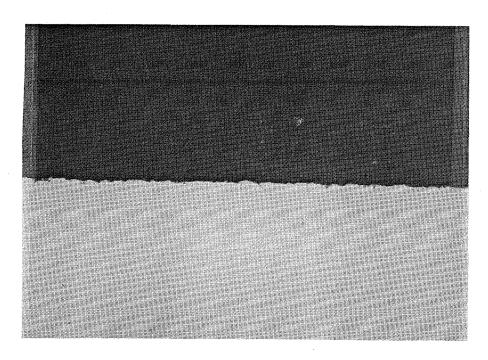


Figure 23. Substrate After 12 Traverses at 45 Volts

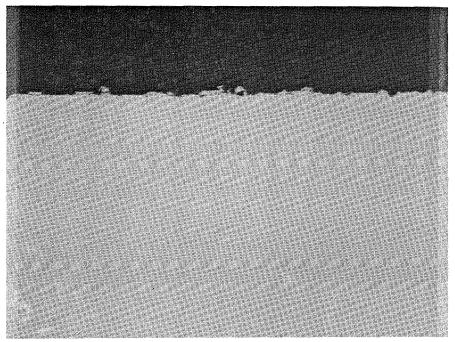


Figure 24. Substrate After 12 Traverses at 50 Volts

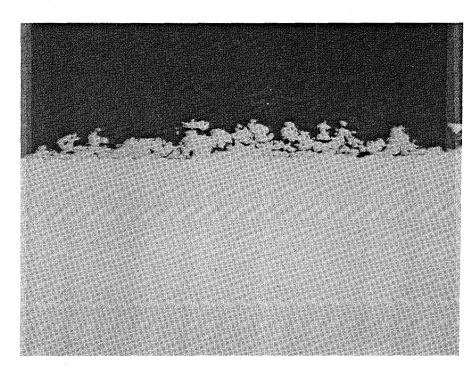


Figure 25. NiCrAlY Deposit After 12 Traverses at 55 Volts

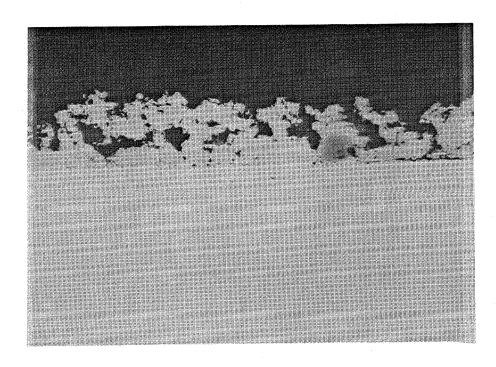


Figure 26. NiCrAlY Deposit After 12 Traverses at 60 Volts

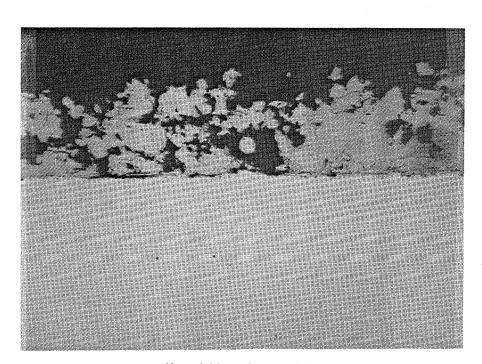
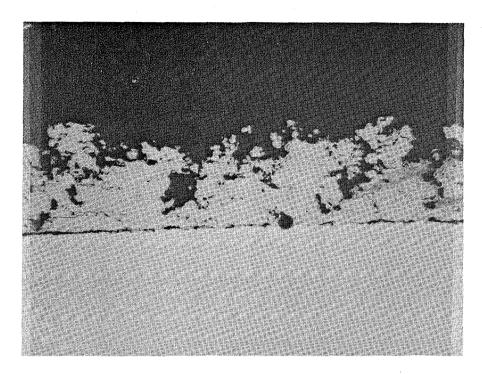


Figure 27. NiCrAlY Deposit After 12 Traverses at 65 Volts



But for Amilia il married

Magnification: 100X

Figure 28. NiCrAly Deposit After 12 Traverses at 70 Volts

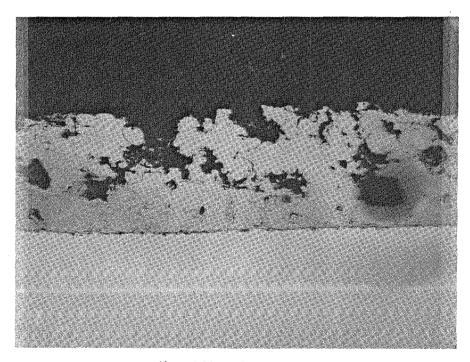


Figure 29. NiCrAlY Deposit Ater 12 Traverses at 75 Volts

At 70 volts, Figure 28, the coating exhibited some densification with large voids. The substrate-coating interface is severely oxidized indicating poor adherence. The slope of the voids and deposit to the right is evidence of the 45 degree spray angle.

The deposit at 75 volts (Fig. 29) is similar to that at 70 volts. Void size has increased but the coating is more uniform, particularly at the bond line.

3.1.4 Effect of Arc Current on Structure

Varying the arc current independently within normal operating parameters had little effect on coating structure. With the equipment configuration used for this test, operating current is limited to the range of 350-500 amperes. The ceramic baseline coating sprayed at 400 amperes was shown in Section 3.1.1. The current was reduced to 350 amperes for the specimen shown in Figure 30. Little, if any, differentiation can be made between it and the baseline coating that is attributable to the lower current. A coating sprayed at 475 amperes (Fig. 31) showed little change in structure. A slight tendency of the voids towards a spherical shape was observed but this could not be established on a reproducible basis. Additional tests with higher currents

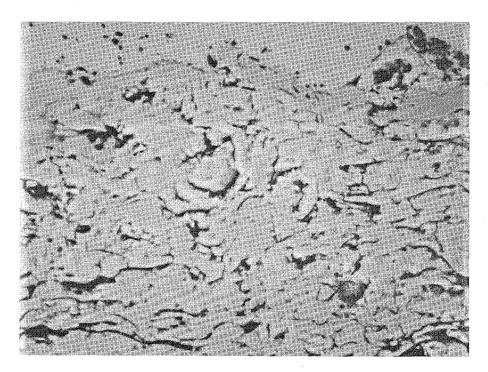


Figure 30. Thermal Barrier Coating With Plasma Arc Current of 350 Amperes

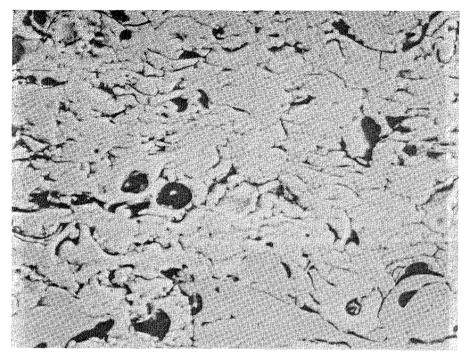


Figure 31. Thermal Barrier Coating With Plasma Arc Current of 475 Amperes

were performed as described separately in Section 3.1.6. These entailed the use of a different anode to allow operation at 80 kW where other modifications to the operating parameters masked the effect of arc current alone.

3.1.5 Anode Selection Effects

The water cooled copper anode used in a plasma spray gun also functions as a nozzle for controlling and ducting the plasma gas flow. Design of this component is one of the primary factors in determining the exit gas velocity and maximum arc current that can be used. The data in Table 7 was supplied by Metco Inc. as a guide for evaluating the performance characteristics of the various anodes available for the 7M system.

Trial coatings were sprayed with two of these anodes using the parameters given in Table 8. Manufacturer recommended parameters were used in each case. A baseline coating deposited with the GH anode is shown in Figure 32. This structure is typical of the thin coatings developed during this study.

The type 700 anode is similar to the GH but is operated at higher power levels. A bond coat deposited using this anode is shown in Figure 33. Coating thickness developed more rapidly using the 700 anode than with the type GH anode. The deposit also evidenced greater porosity and oxidation than the baseline (GH anode) coating. No obvious benefit in coating quality was observed with

Table 7
Plasma Spray Anode Performance Characteristics

			num Elec Paramete		Plasma	Nozzle
Anode	Gas System	kW	Amps	Volts	Exit Velocity	Life (hrs)
GH 700 703 704	A/H ₂ A/H ₂ A/H ₂ A	37 42 45 58	500 600 600 1000	75 70 75 58	1829 1768 2621 2835	25 20 15 30

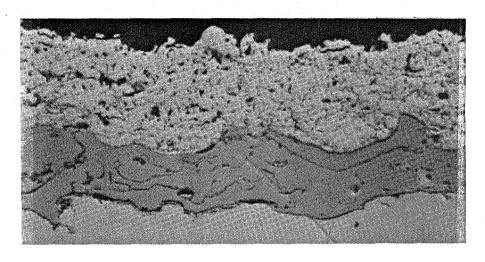
Table 8
Effect of Anode Selection

Coating	Anode	Current (amperes)	Voltage (volts)	Power kW	Standoff (cm)	Thickness (mm)
Bond	GH	500	65	32	10.0	0.025
Ceramic	GH	450	50	22	7.6	0.025
Bond	GH	400	50	20	10.0	0.051
Ceramic	704	1000	54	54	7.6	0.025
Bond	704	1000	52	52	10.0	0.130

this anode. Operating the 700 anode at lower power levels produced coatings similar to those obtained with the type GH anode.

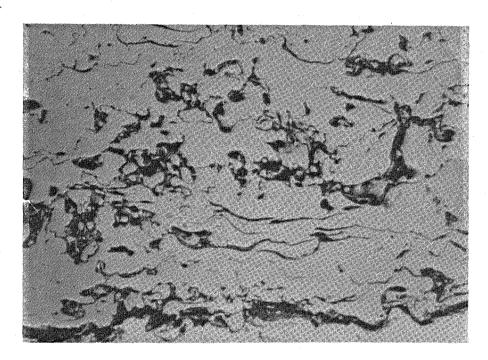
The type 703 anode operated at 45 kW produced excessive substrate oxidation as shown in Figure 34. Severe oxidation pits are visible in the copper substrate, and unmelted particles are evident where particle melting and vacuum fusion of the NiCrAly produced entrapped voids. The laminar structure normally seen in these structures is not evident.

Two examples of coatings applied with the type 704 anode are shown in Figures 35 and 36. A zirconia coating applied over a standard bond coat is shown in Figure 35, and Figure 36 shows just a NiCrAly bond coat applied with the type 704 anode. In both tests the deposit rate was low, and thin dense coatings resulted. Severe oxidation occurred at the bond coat-substrate interface, indicating overheating. The bond coat applied under these conditions was also rich in oxides. No indications of coating improvment were obtained with these special anodes and it was decided to continue tests with the standard GH anode.



Magnification: 500X

Figure 32. Baseline Coating Applied with a GH Anode



Magnification: 500X

Figure 33. Bond Coat Deposited at 42 kW With A Type 700 Anode

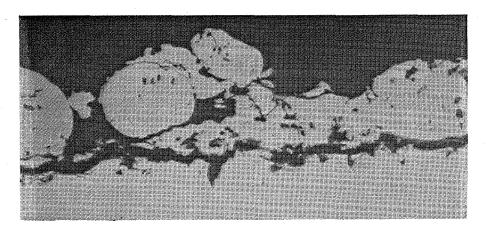


Figure 34. Bond Coat Applied at 45 kw With a Type 703 Anode

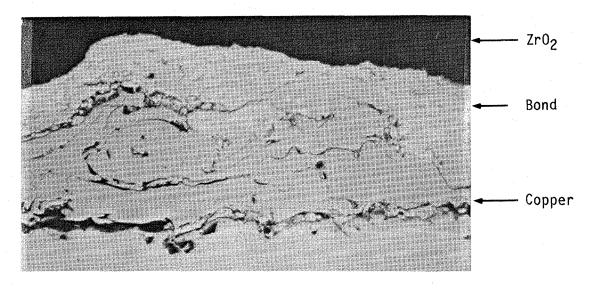


Figure 35. Zirconia Applied to the Baseline Bond Coat With a 704 Anode at $58\ kW$

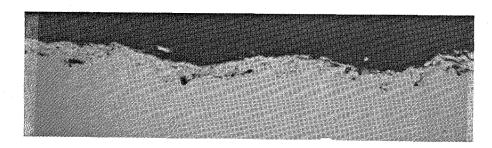


Figure 36. Bond Coat Applied With a Type 704 Anode at 58 kw

3.1.6 Effect of Traverse Rate

The baseline traverse rate used in these tests was 2.5 cm/sec. This was reduced incrementally to 1.2 cm/s. No visible change was found in the coating microstructure. However, as traverse rate was decreased, substrate temperature increased. At 1.2 cm/s this increase amounted to 12°C. This slight increase in substrate temperature did not visibly affect bond quality.

3.1.7 Powder Feed Rate

Zirconia feed rate was varied over the range of 0.04 to 0.09 gm/s. The baseline value was 0.05 gm/sec. Reproducible feed rates lower than 0.04 gm/s could not be achieved. The reduction in feed rate below baseline (Fig. 37) did not affect the coating microstructure. As feed rates were increased, laminar voids developed as shown in Figures 38 and 39. At feed rates higher than 0.09 gm/s deposit efficiency decreased, as evidenced by visible, unmelted particles bouncing off the fixture and substrate. Generally, it was found that the lower the feed rate the more uniform the coating, with a corresponding reduction in unmelted particles in the coating.

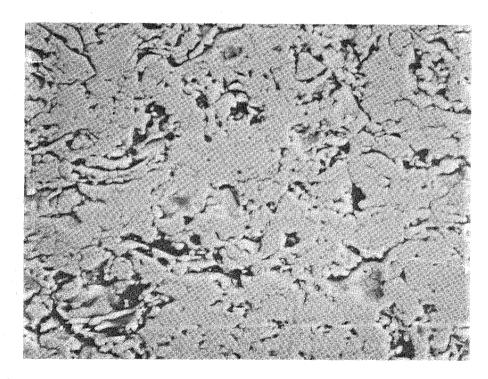


Figure 37. Thermal Barrier Coating With Powder Feed Rate of 0.04 gms/sec

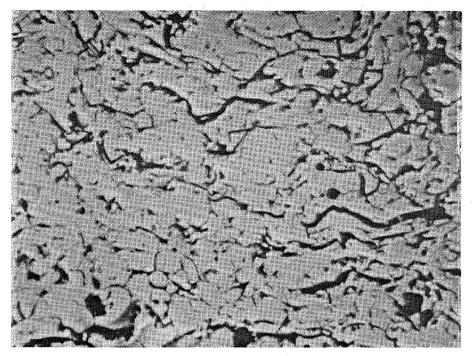


Figure 38. Thermal Barrier Coating With Powder Feed RAte of 0.062 gms/sec

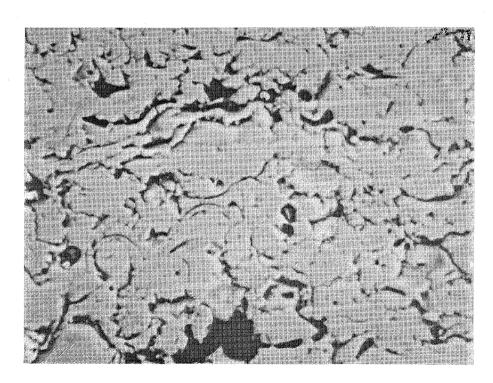


Figure 39. Thermal Barrier Coating With Powder Feed RAte of 0.09 gms/sec

3.2 PROPERTY MEASUREMENT

Based on the laboratory data from the studies performed in Section 2, three sets of ceramics and one set of bond coat deposition parameters were selected for further study. The parameters are given in Table 9. Examples of coated specimens used for property measurement testing are shown in Appendix 3.

Table 9
Plasma Spray Parameters Selected for Coating Property Measurements

	Ceramic Coating #1	Ceramic Coating #2	Ceramic Coating #3	Bond Coat
Current (amps) Voltage (volts) Power (kW) Argon (m³/hr) Hydrogen (m³/hr) Stand-Off (cm) Plasma Velocity (m/s) Powder Port Cooling Air (kPa) Nozzle	400 50 20 2.97 0.085 6.4 25.4 No. 2 551 GH	500 65 32 2.97 0.14 6.4 30.5 No. 2 551 GH	600 70 42 4.25 0.20 6.4 43.7 No. 3 551 703	500 65 32 5.66 0.11 32.5 No. 2 551 GH

The same bond coat parameters were used in all tests to apply a nominal 0.025 mm coating to the copper substrate. Substrate temperatures were held within the range of 97 to 120°C during coating application. Higher substrate temperatures caused excessive oxide formation at the bond coat-copper interface. The first system (Fig. 40) used the standard bond coat and the zirconia-8% yttria was applied at 20 kW. The 32 kW coating is shown is Figure 41 and the 40 kW coating is shown in Figure 42.

As the power level increased the coatings exhibited a reduction in pore size and an increase in porosity. It was found that the coatings produced at 20 and 40 kW could be duplicated readily. However, those produced at 32 kW varied in density and other properties as discussed in Section 3.5. Generally, as anode wear increased the coating tended to become less dense. This effect was only evident on the 32 kW coatings and no quantitative relationship could be established. Anode usage was limited to four hours to minimize anode wear effects but they could not be eliminated from the 32 kW coatings. Because of this variation, the property data reported in this section is limited to values obtained from specimens produced with new anodes unless otherwise noted.

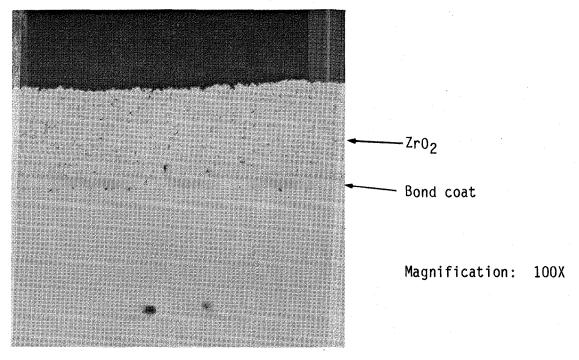


Figure 40. Thermal Barrier Coating Applied at 20 kW (System 1)

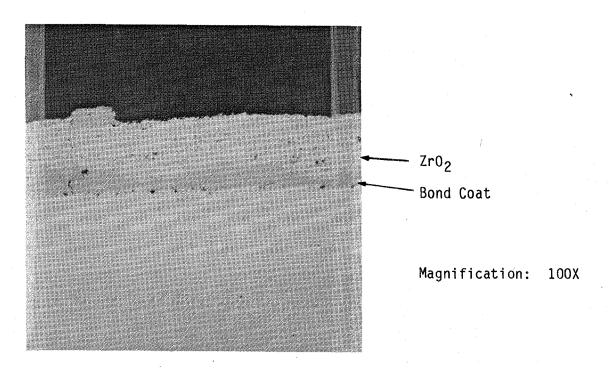


Figure 41. Thermal Barrier Coating Applied at 32 kw

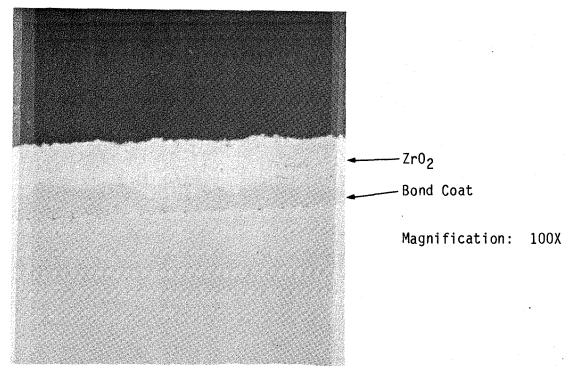


Figure 42. Thermal Barrier Coating Applied at 40 kw

3.2.1 Density

Coating density was determined by three methods:

- . Visual inspection of a 100X photomicrograph
- Photometric evaluation of 50X photomicrographs
- . Weighing and measuring a sample of the coating

The following average values were obtained:

Zr02.8Y203 Density Values

Visual (% Theory)	Photometric (% Theory)	Weight (g/cc)
85	80	4.8
92	90	5.2
95	95	5.3
	(% Theory) 85 92	(% Theory) (% Theory) 85 80 92 90

3.2.2 Thickness Control

Because of the extreme thinness of the coatings required in this program (0.013 to 0.10 mm) control of the application parameters was considered an important factor. Conventional coatings vary by ± 0.02 mm or more from the nominal thickness. For the coatings evaluated in this study, this could amount to $\pm 12\%$ of the thickest coatings while producing completely uncoated areas with the thinner coatings. To assure a precise and uniform coating, it was found that low powder feedrates, in the range of 0.05 to 0.10 gm/s, were required. Under these conditions typical build-up rates were 0.0025 to 0.008 mm per traverse. The low powder feed also provided a more uniform ceramic deposit, free of entrapped particles. Under these conditions coatings 0.025 mm thick could be produced to an accuracy of ± 0.005 mm.

3.2.3 Thermal Diffusivity and Conductivity

Thermal conductivity and diffusivity of the coatings were measured as described previously. The values obtained are reported in Table 10 and in Figures 43 and 44.

The coatings applied at 20 and 40 kW show an increase in thermal conductivity with increased gun power. The 32 kW coating is inconsistent in that it does not lie between the other two values as would be anticipated. In other tests the 32 kW coating responded as if it were less dense than the 20 kW coating. The source of this erratic performance at 32 kW has not been identified. Apparently at this power level the plasma flame is sensitive to minor variations in anode wear and gas flow, at levels below the sensitiity of the process controls. Because of this fluctuation, the 20 and 40 kW coatings were selected for coating the experimental thrust chambers.

3.2.4 Surface Finish

Surface finish did not appear to be a function of plasma spray gun parameters. When surface finish values of the as-sprayed coatings were checked, arithmetic average values of 5-10 micrometers were obtained. Generally these values appeared to be the result of initial particle size rather than power levels, once a uniform coating was obtained.

3.2.5 Crystal Structure

X-ray diffraction measurements of the coatings gave nominal values of 94-98% cubic and tetragonal with the remainder being monoclinic. The initial powder was reported as 10% monoclinic with the balance cubic. This transformation of the monoclinic zirconia to cubic/tetragonal was similar in all cases. There appeared to be less monoclinic zirconia (about 3%) in the 20 kW coatings

 $\label{thm:table 10} Table \ 10$ Thermal Diffusivity and Conductivity of $\rm Zr0_2Y_2O_3$ Layers

Sample	Temperature (°C)	Thickness (cm)	Density (gm cm ⁻³)	Specific Heat (W s gm ⁻¹ K ⁻¹)	Half-Time (sec)	Diffusivity (cm ² sec ⁻¹)	Conductivity (W cm ⁻¹ K ⁻¹)	Conductivity (Btu in hr ⁻¹ ft ⁻² F ⁻¹)
20 kW	23	0.0254	4.8	0.457	0.0775	0.00298	0.00654	4.53
	100	0.0254	4.8	0.491	0.0891	0.00250	0.00590	4.09
	200	0.0254	4.8	0.522	0.0924	0.00238	0.00596	4.13
	300	0.0254	4.8	0.541	0.0965	0.00229	0.00593	4.11
	400	0.0254	4.8	0.552	0.0101	0.00219	0.00581	4.03
	500	0.0254	4.8	0.567	0.1009	0.00218	0.00591	4.10
32 kW	23	0.0178	5.2	0.457	0.0337	0.00413	0.00981	6.80
	100	0.0178	5.2	0.491	0.0338	0.00397	0.01010	7.00
	200	0.0178	5.2	0.522	0.0354	0.00372	0.01010	7.00
	300	0.0178	5.2	0.541	0.0365	0.00363	0.01020	7.07
	400	0.0178	5.2	0.552	0.0380	0.00345	0.00991	6.81
	500	0.0178	5.2	0.567	0.0383	0.00342	0.01910	7.00
40 kW	23	0.0178	5.2	0.457	0.0339	0.00379	0.00901	6.25
	100	0.0178	5.2	0.491	0.0353	0.00356	0.00909	6.30
	200	0.0178	5.2	0.522	0.0384	0.00321	0.00871	6.04
	300	0.0178	5.2	0.541	0.0397	0.00311	0.00875	6.07
	400	0.0178	5.2	0.552	0.0408	0.00302	0.00866	6.00
	500	0.0178	5.2	0.557	0.0421	0.00292	0.00860	5.96

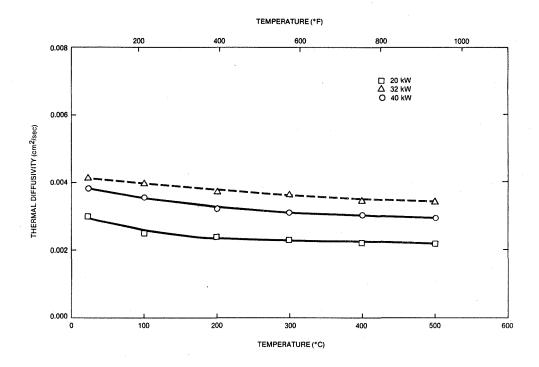


Figure 43. Thermal Diffusivity of ZrO₂Y₂O₃ Layers

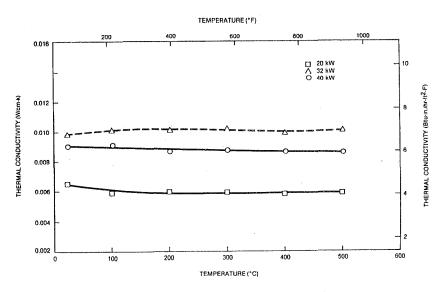


Figure 44. Thermal Conductivity of ZrO₂Y₂O₃ Layers

than in the 40 kW coating, but this variation was within the error limits of the analysis and it could not verified.

3.2.6 Calorimeter Tests

The 1.27 cm calorimeter probe is shown in Figure 45 and the test surface installed in a guard is shown in Figure 46. Some loss of insulation between the probe and guard is visible in the second figure. When this loss occurred the change in outlet temperature was less than $0.5\,^{\circ}\text{C}$ and the results remained within experimental accuracy.

In operation the plasma flame was directed at the 1.27 cm diameter face of the calorimeter. Water flow and the T between the inlet and outlet water

temperatures were recorded for calculating the total heat input to the cold face calorimeter. Both coated and uncoated calorimeters were tested.

The appearance of a zirconia coated calorimeter is shown in Figure 47. The coating was applied to a thickness of 0.10 mm at 40 kW. In this test, the calorimeter was exposed to a 32 kW plasma flame. The relationship between power level and gun distance to heat flux is shown in Figure 48. Input to the calorimeter was measured for three sets of gun parameters as shown, and plotted as a function of distance. The gun parameters were given previously in Table 9.

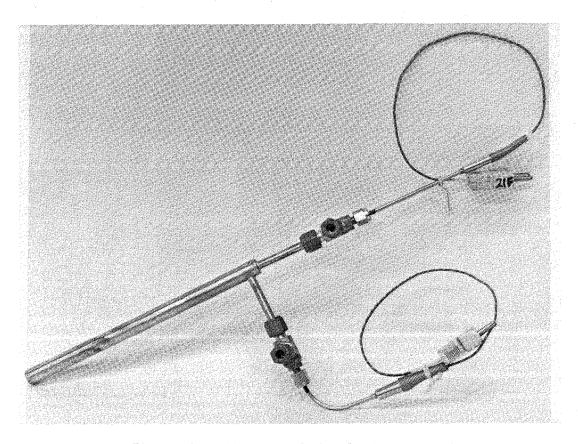


Figure 45. Water Cooled Calorimeter Probe

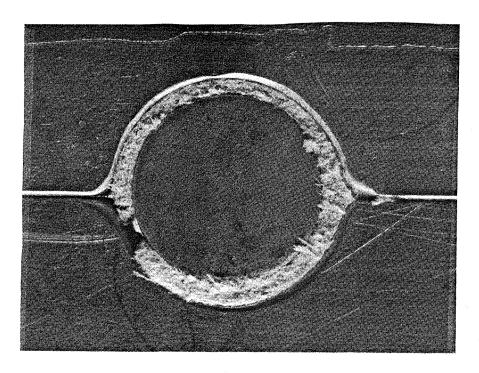


Figure 46. Calorimeter Face in Guard Shield

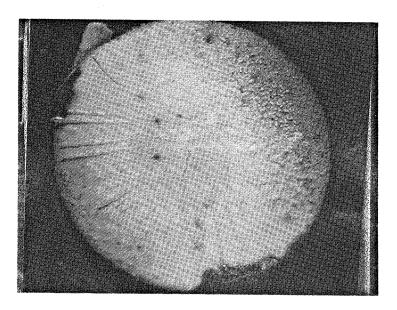


Figure 47. Calorimeter Face With 40 kW $\rm Zr0_2Y_2O_3$ Coating 0.10 mm Thick Exposed to a Heat Flux of 9.15 watts/mm²

Heating appears to be more sensitivite to voltage (hydrogen flow) than to current. The current increased by 100 amperes for each increase in power. The 15 volt increase between 20 and 32 kW shows a far greater increase in heat input than the 5 volt increase between 32 and 40 kW.

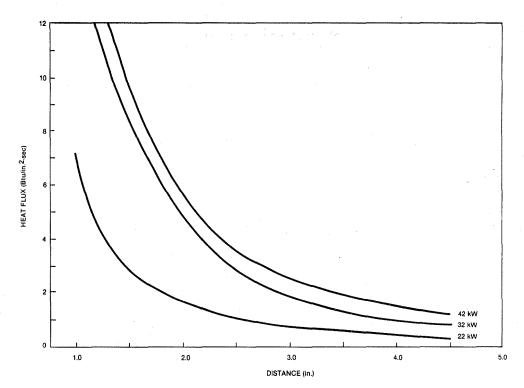


Figure 48. Heat Flux From a Plasma Torch as a Function of Distance at Selected Power Levels

The effect of the zirconia coatings on heat input to the calorimeter is reported in Figure 49 through 51. The 20 kW coating provided the greatest reduction in heat input to the calorimeter. The coating remained intact until the heat flux through the coating reached 8.2 watts/mm² at which point surface melting occurred. When the 32 kW coating was tested failure occurred at a heat input of 9.0 watts/mm² through spallation of the coating. The third coating, 40 kW zirconia, survived until surface melting occurred at an input of approximately 16.3 watts/mm² or twice that of the 20 kW coating. Failure of this coating also occurred as a result of melting. Some cracking normal to the surface occurred that may also have contributed to the low insulative value obtained with this coating.

Generally, the tests confirmed the thermal conductivity values previously measured and showed that the coatings could be retained up to the point of surface melting. The poor performance of the 32 kW coating supported the selection of the 20 and 40 kW for the coating of test thrust chambers.

3.2.7 Residual Stress and Stress Free Temperature

Several flat copper strip specimens were coated for residual stress and stress free temperature measurement. A copper thickness of 0.8128 mm and a bond coat thickness of 0.0508 mm were used for these tests. Density, residual stress

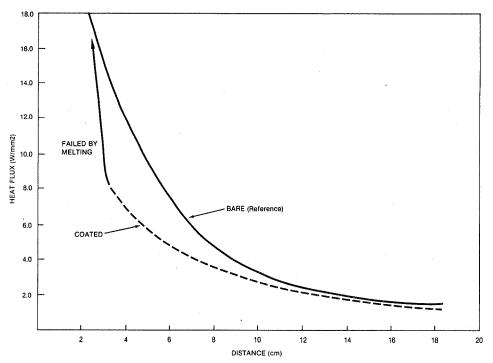


Figure 49. Heat Flux Transmitted to a Copper Calorimeter at 32 kW Through a 0.1 mm 20 kW Zirconia Coating

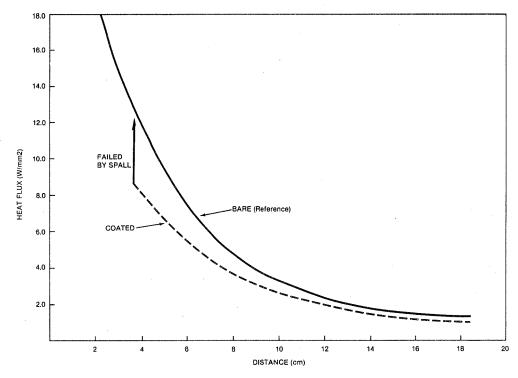


Figure 50. Heat Flux Transmitted to a Copper Caloriometer at 32 kW Through a 0.10 mm, 32 kW Zirconia Coating

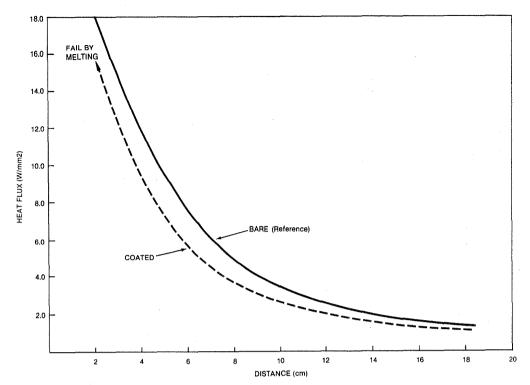


Figure 51. Heat Flux Transmitted to a Copper Calorimeter at 32 kW Through a 0.10 mm, 40 kW Zirconia Coating

and stress free temperature (SFT) were measured as described in the previous section. The results are shown in Table 1. The spray parameters for these coatings are given in Table 9.

These data show an increase in density and a rather constant residual stress and SFT with an increase in gun power from 20 kW to 32 kW. A substantial increase in residual stress and SFT was observed when the gun power was increased further, to 42 kW, although the desnsity remained constant. Increasing the coating thickness from 0.0965 mm to 0.254 mm also caused a significant increase in residual stress and SFT. No density measurement was made on this latter coating.

It is interesting to note that these data show the coating to incur a compressive residual stress during plasma spraying. This could effectively weaken the coating in service since compressive spallation is the predominant coating failure mode. However, the data generated thus far is rather limited and does not warrant conclusive interpretation.

The coatings produced in this study were strength tested in their as-sprayed condition (e.g., no annealing). Thus, the strength data generated reflects any effect of residual stress on coating strength. Reduction of residual stresses may present a practical means of increasing coating strength. This concept, although not pursued on this program, may be worthy of future coating development studies.

Table 11

Residual Stress and Stress Free Temperature Results From Coated Flat Specimens (0.8128 mm copper and 0.0508 mm NiCrAly bond)

Coatir Type*	•, •	Weight Gain (gms)	δ (mm)	α ₁	 L (mm)	S (MPa)	T _{SF}	Density
I	0.1 0.1	0.3990 0.3778	0.1778 0.3048	0.09652 0.09652	50.8 50.8	-7.2 -12.3	58 84	88.6 84.0
II	0.6 0.6	0.4341 0.4315	0.2540 0.2794 5.4864	0.09652 0.09652 0.02540	50.8 50.8 55.12	-10.3 -11.3 -60.2	74 79 447	96.5 96.0
III .	0.4	0.5209	1.9812	0.1016	50.8	-75.6	437	93.0

3.3 ANALYTICAL MODELING

3.3.1 Thrust Chamber Model Development

Initial finite element performance analyses of the thrust chamber coating system were subcontracted to Control Data Corporation. This initial work included the model construction, preliminary test runs, and several coating analyses. All later finite element analyses were performed at Solar.

The original thrust chamber coating system model was 50.8 mm long and was divided into six elements axially. This resulted in an element aspect ratio (length/width) of 667 for the bond coat elements, which are the most extreme. The model length was later changed to 0.229 mm, which reduced the bond coat element aspect ratio to 3. This change caused no difference in the thermal analyses and less than a three percent variation in the calculated stresses. The shorter model was used for all of the final coating analyses.

The finite element model of the thrust chamber coating system was used to evaluate typical coating performance under selected operating conditions. Material properties from the literature were used in these analyses (Table 1). The selected operating conditions included a hot ceramic surface temperature of 1650° C (the maximum desirable ceramic operating temperature), cold copper surface temperatures of -240, -18 and 204° C, and a specified heat flux of 18 W/mm². These temperature selections were explained earlier but this heat flux was selected inadvertently due to an error in the original analyses performed by CDC. The heat flux was originally intended to be 49 W/mm² but the units were misinterpreted to be W rather than W/mm². Nonetheless, these

analyses were useful in developing the coating model and observing the general behavior of the coating system under potential operating conditions.

These initial coating systems analyses used a constant ceramic coating thickness of 0.10 mm. Steady state analyses were performed to evaluate the temperature distributions, heat fluxes and stress profiles through the coatings.

One series of coating system analyses used a constant hot surface temperature of 1650°C and three cold side temperatures: -240, -18 and 204°C. The resulting temperature gradients are shown in Figure 52. This figure shows that the bulk of the temperature drop occurs through the thin ceramic layer, subjecting the copper to a relatively low temperature gradient. The temperature gradients through the coating and the copper do not appear to be significantly influenced by the cold side temperature, although the overall temperature difference was increased by 31% from the smallest value to the largest.

The stress profiles resulting from the temperature gradients shown in Figure 52 are presented next. The hoop (or circumferential) stresses are equal to the axial stresses (due to the radial temperature gradient) and are shown in Figure 53. The radial stress profiles are shown in Figure 54. Step discontinuities in these stress profiles occur at the interfaces of different materials in the coating system. These discontinuities result from the changes in elastic moduli between materials, since the analyses assume continuous strain across the interfaces.

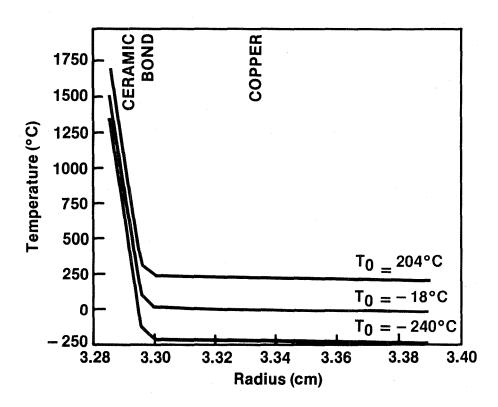


Figure 52. Temperature Radient for 1650°C Wall Temperature

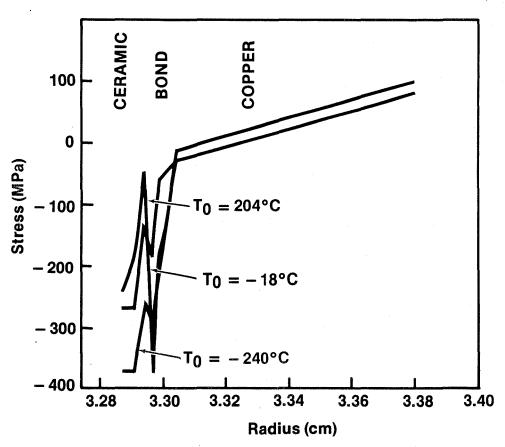


Figure 53. Axial Stress Gradient for 1650°F Wall Temperature

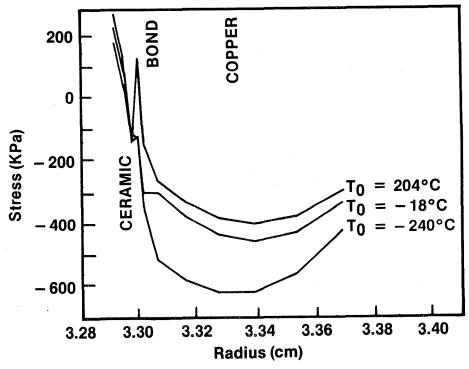


Figure 54. Thrust Chamber Radial Stress Gradient for 1650°C Inner Wall Temperature

The hoop and axial stresses in the coating are shown to be compressive; the maximum values occurring at the coating surface with a linear reduction through the thickness. The stresses in the bond coat are again compressive, while the stresses in the copper change from compressive to tensile. The coating stresses are of significant magnitude, with the maximum stresses on the order of -240 to -400 MPa. Stresses in the copper wall range from -31 to +110 MPa. The maximum tensile stress in the copper falls between the yield strengths of annealed and hardened oxygen-free copper.

The radial stresses through the coating system are much less severe than the hoop and axial stresses. Thus, the coating system is essentially in a biaxial stress state during operation. Radial stresses in the ceramic are approximately 0.5 MPa maximum, while the radial stresses in the copper are on the order of -0.9 to -1.6 MPa, maximum.

The cold surface temperature appears to have a significant effect on the hoop and axial stresses in the coating. A cold temperature change from 204 to -240°C (a 31% increase in overall temperature differential) results in a 60% increase in the maximum compressive coating stress. The effect on the hoop and axial stresses in the copper is less pronounced, with approximately a 45% increase in stress observed for the same change in cold side temperature. Cold side temperature also had an effect on the radial stresses, but the stress magnitude remained relatively insignificant.

A second set of coating system analyses used a constant radial heat flux of 18 W/mm² through the coating and the same three cold surface temperatures as before: -240, -18 and 204°C. The resulting temperature gradients are shown in Figure 55. The heat flux specified for these analyses was similar to the resulting heat fluxes in the previous analyses. In fact, the two cases having a 204°C cold side temperature are nearly identical.

As expected, the hot surface temperature decreases with decreasing cold surface temperature, since the heat flux is constant. It is interesting to note that the decrease in hot surface temperature (333°C) is approximately 25% less than the decrease in cold surface temperature (444°C), over the range evaluated. This is attributed to temperature affects on the material properties. As in the previous analyses, most of the temperature drop occurs through the ceramic coating, subjecting the bond and substrate to to relatively small thermal gradients.

The hoop and axial stress profiles resulting from the temperature gradients in Figure 55 are shown in Figure 56. These stress profiles are quite similar to those of the previous analyses, again showing the ceramic coating under very high compressive stress. The copper is subjected to slight compressive stresses at the bond interface and moderate tensile stresses at the outer surface.

These first two series of analyses were conducted to exercise the thrust chamber coating system model, and to evaluate the general behavior of the coating system under typical rocket engine operating conditions. The model was found to operate acceptably and the coating system behavior was found to be reasonable.

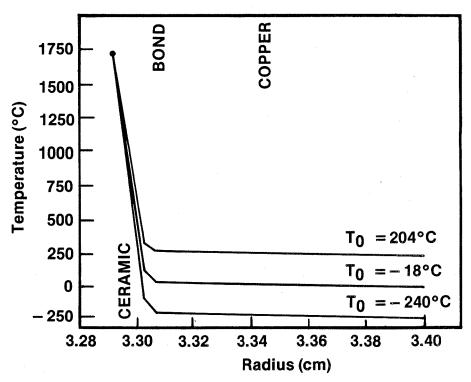


Figure 55. Thrust Chamber Temperature Gradient for 1650°C Inner Wall Temperature

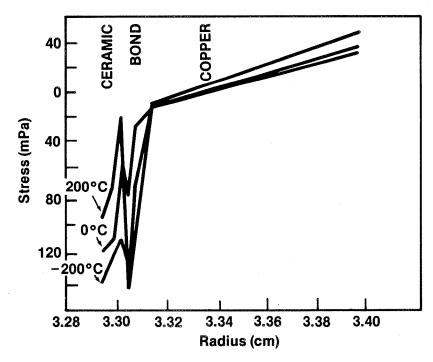


Figure 56. Thrust Chamber Axial Stress Gradient for 1650°C Inner Wall Temperature

This model was then used to evaluate the effect of thickness on the predicted coating temperatures and stresses in the thrust chamber environment. These analyses again used literature values for material properties. The purpose of these analyses was to observe the general effect of coating thickness on coating stress and temperature, and to establish a probable range of the coating thickness required for rocket thrust chamber application.

Figure 57 shows both surface temperature (left scale) and hoop (or axial) stress (right scale) in the coating for a constant heat flux (20 w/mm^2). This heat flux is at the lower end of a typical thrust chamber operating range. Both coating temperature and stress increase linearly with coating thickness.

This analysis indicates that a coating thickness of less than 0.13 mm may be required to maintain an acceptable coating temperature (2000° C) at this heat flux condition. The compressive coating stresses in this thickness range would be less than 400 MPa. The significance of this was not known at the time of these analyses.

Figure 58 shows the effect of coating thickness on heat flux for a fixed temperature gradient. The hot surface temperature (1650° C) used for this analysis is a desirable upper limit for the coating. The cold temperature (-18° C) is a reasonable anticipated operating value. This curve shows the heat flux is inversely proportional to coating thickness. At very low thicknesses, the heat flux is very sensitive to coating thickness. Again, the

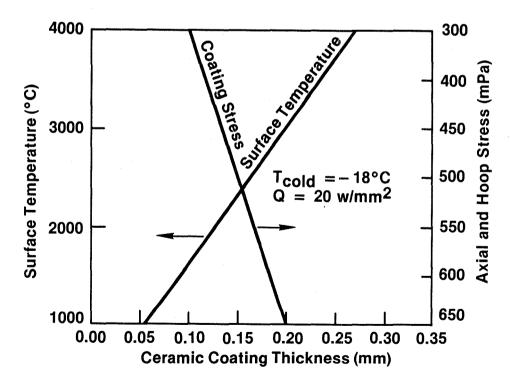


Figure 57. Relationship of Coating Surface Temperature and Stress to Coating Thickness to Constant Heat Flux

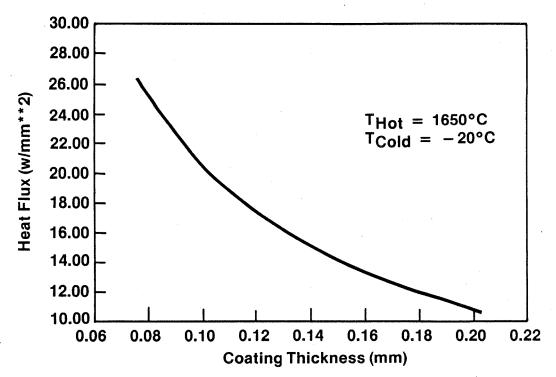


Figure 58. Initial Predicted Heat Flux Versus Coating Thickness at Constant ΔT

range of heat fluxes covered by this graph is in the moderate operating regime. This analysis indicates that coating thicknesses of less than 0.075 mm may be required to accommodate the high heat flux conditions without excessive temperature. The maximum compressive coating stress for this case is constant with varying coating thickness and is 350 MPa.

3.3.2 Coating Structure Model Development

Finite Element Model of Porous Coating

A variety of coating structures were evaluated using the finite element model shown in Figure 5. These structures included a variety of pore dimension ratios (D_r) , total porosities (P_t) , and pore sizes (P_i) . An attempt was made to establish a relationship between these three coating parameters (D_r, P_t, P_i) and the effective thermal conductivity of the coating system (k).

An evaluation of the significance of radiation to the overall heat flow through a TBC was conducted. This was accomplished by comparing the heat flux through one coating model, with and without radiation across the pores. A negligible effect was observed. Thus, radiation was omitted from further analyses, to reduce the required computer time.

Figure 59 shows plots of effective thermal conductivity vs. pore dimension ratio for various combinations of total porosity and individual pore size.

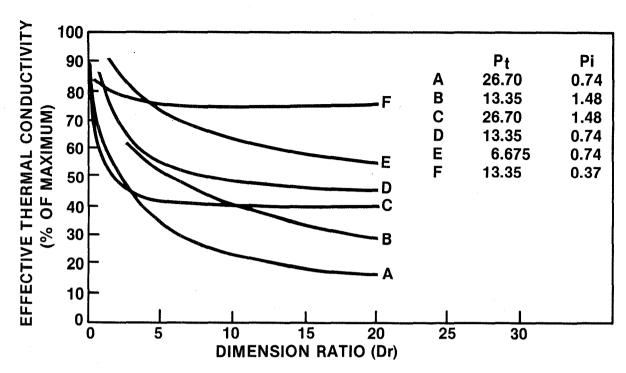


Figure 59. Influence of Microstructure on Effective Thermal Conductivity

Inspection of these curves shows a consistent trend towards higher thermal conductivity with decreasing dimension ratio (D_r) . This trend is most pronounced for small values of D_r and tends to level off at large values of D_r . Also evident from this figure are inconsistencies in the effects of both porosity and pore size on effective conductivity. For example, curves A, D and E show a consistent pattern of increasing conductivity with decreasing porosity. (This trend is intuitively understandable.) However, curves B and C depict the opposite effect; that is, a decreasing conductivity with decreasing porosity. Similarly, curves B, D and F indicate an increase in conductivity with decreasing pore size, while curves A and C show the opposite effect.

Numerical regression analyses were performed to establish mathematical expressions for effective thermal conductivity in terms of one and two coating parameters. Due to the inconsistencies just described, it was not possible at this point to establish expressions containing all three coating parameters. It is believed that different coating parameters must be established, to replace or compliment the three parameters initially selected, in order to adequately and uniquely characterize a coating structure. Limited regression analyses were performed however, to illustrate the approach and methodology employed to establish an analytical thermal model of a porous coating structure.

All of the curves in Figure 59 appear to have an approximately exponential form. One-variable regression analyses were performed to establish mathematical expressions for curves A, D and E. Curves A and E were found to best fit an exponential relationship, while curve D was found to best fit an

inverse D_r (D_r being the independent variable) relationship. Curve D can also be expressed in exponential form with slightly less accuracy. These expressions and their respective correlation coefficients (r) are presented here:

Curve A
$$k = 56.94 - 13.82 \ln (D_r)$$
 $(r = 0.996)$ (10)

Curve E
$$k = 94.13 - 13.26 \ln (D_r)$$
 $(r = 1.000)$ (11)

Curve D
$$k = 67.46 - 7.74 \ln (D_r)$$
 $(r = 0.975)$ (12)

$$k = (42.59 D_r + 59.35)/D_r \quad (r = 0.999) \tag{13}$$

Two-variable regression analyses were performed on two sets of curves in Figure 59. Each set used dimension ratio as one variable. The first set showed a consistent decrease in conductivity with increasing porosity and contained curves A, D and E. The second set showed a consistent decrease in conductivity with increasing pore size, and contained curves B, D and F. The following expressions were established for k:

$$k = 526.8 P_t^{-0.70} D_r^{-0.32} \quad (r_m = 0.906)$$
 (14)

for a pore size of $P_i = 0.74$ and

$$k = 65.4 P_i^{-0.31} D_r^{-0.13} \qquad (r_m = 0.767)$$
 (15)

for a porosity of P_t = 13.35; r_m is the multiple regression correlation coefficient.

Some of the analytical "data" generated using the finite element porous coating model were used to create the plot of effective thermal conductivity versus porosity shown in Figure 60. All of the curves presented show the same general trend of decreasing thermal conductivity with increasing porosity, except for curve G, which shows a slight rise in conductivity when the porosity is increased from 15 to 25 percent. It should be noted that curve G still lies in the range of "scatter" of the other curves.

Curiously, curve G also most closely matches the experimental data, plotted as curve H, for porosities less than 15 percent. The experimental data is derived from measured values of coating conductivities and porosities (reported earlier) and an assumed value for the thermal conductivity of the solid ceramic of 2.06 W/mC, which is one of the commonly reported literature values. It appears from Figure 60 that the finite element model of a porous coating generally predicts the same effect of porosity on thermal conductivity as observed experimentally. Although the experimental data falls outside of the analytical results, the shape of the curve is basically consistent. The objective of the analytical model was to simulate the effects of coating structure parameters on thermal conductivity, and then to empirically adjust the model to accurately calculate the actual value of a coating's thermal conductivity.

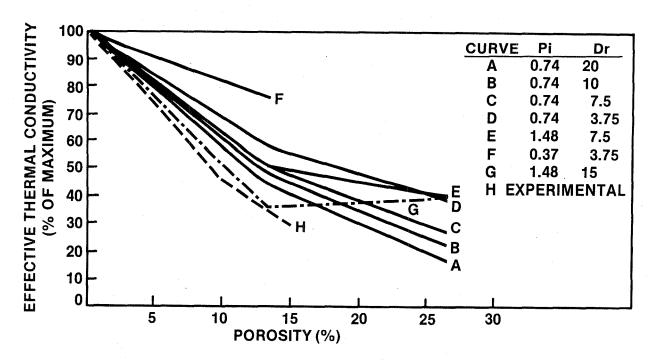


Figure 60. Finite Element Analysis of Porosity Effect on Effective Thermal Conductivity of Coatings

Due to the inconsistencies in the analytical model results described earlier, no attempt to empirically adjust the model was made. The results thus far support the contention that such a model can be empirically modified to produce accurate results. However, the basic model must first accurately represent all of the critical structural parameters of a porous coating. Therefore, additional work is required to establish more appropriate coating characterization parameters before a useful analytical model can be developed.

Theoretical Thermal Conductivity Predictions for Porous Coatings

Much work has been done to theoretically predict the properties of mixtures of materials (Refs. 6, 7). This work encompasses mixtures of all numbers and types of phases. All binary mixtures can be divided into three basic classes: (1) one-phase miscible mixtures, (2) two-phase systems with one continuous phase and one dispersed phase and (3) two-phase systems with two continuous phases. There are general mixture rules for each of these classes of mixtures for predicting material properties. It is most often necessary to combine empirical data with theoretical equations to accurately predict specific properties of a particular mixture.

A porous ceramic coating may be characterized as a mixture with one solid continuous phase (the ceramic) and one gaseous dispersed phase (the pores). In this case, the continuous phase is considered to be the "hard" phase and the dispersed phase is the "soft" phase. In addition, the continuous phase in a porous ceramic is most likely the major phase; that is, the phase com-

prising the largest volume fraction. Figure 61 illustrates this type of mixture.

The thermal conductivity of mixtures such as that depicted in Figure 61 have been evaluated by Maxwell and applied by Eucken (Ref. 6). Eucken suggests an expression for the resultant thermal conductivity of a mixture (k_m) as

$$k_{m} = k_{c} \left[\frac{1 + 2 v_{d} (1 - k_{c}/k_{d})/(2k_{c}/k_{d} + 1)}{1 - v_{d} (1 - k_{c}/k_{d})/(k_{c}/k_{d} + 1)} \right]$$
(16)

where k is the thermal conductivity, v is the volume fraction and the subscripts c and d stand for the continuous and dispersed phases, respectively. When $k_c \gg k_d$, the expression for resultant conductivity can be simplified to

$$k_{\rm m} = k_{\rm c} \left[(1 - v_{\rm d})/(1 + v_{\rm d}) \right]$$
 (17)

and if $k_d \gg k_c$ the expression simplifies to

$$k_{\rm m} \approx k_{\rm c} \left[(1 + 2v_{\rm d})/(1 - v_{\rm d}) \right]$$
 (18)

Note that Equation (18) expresses the resultant (or effective) thermal conductivity of a mixture solely in terms of the conductivities and volume fractions of each component in the mixture. There is evidence to suggest that other factors are of equal importance in determining effective conductivity (Ref. 7). For example, some two-phase systems, such as Al_2O_3 - ZrO_3 and MgO- $MgAl_2O_4$, can exhibit thermal conductivities lower than either single phase. This results from flat microcracks which open along grain boundaries during thermal cycling due to differences in the thermal expansion coefficient within the material. The small fraction of porosity which results from this cracking may

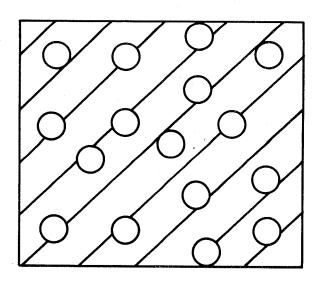


Figure 61. Illustration of Mixture Having a Continuous Major Phase and Dispersed Minor Phase

cause a significant reduction in thermal conductivity when the porosity occurs as a continuous phase perpendicular to the heat flow path. Similarly, the thermal conductivities of other oxides can be significantly altered by heat treating, with little change in porosity or grain size.

Thermal cycling of some two-phase solid mixtures has been observed to continuously change the effective thermal conductivity due to crack growth during cooling and crack annealing during heating. The opening and closing of such cracks have caused hysteresis in conductivity measurements of some materials (e.g., aluminum titanate) (Ref. 7). This type of effect from thermal cycling may be pertinent to the behavior of thermal barrier coatings, especially those subjected to high heat fluxes. No reference of this behavior has been found for TBC's in particular, but the phenomena is worthy of note.

Temperature may also influence the thermal conductivity of a mixture by affecting the conductivity of each phase. In the case of pores containing air, not only does the true thermal conductivity of air increase sharply with temperature, but so does the effective thermal conductivity due to radiation heat transfer. This is illustrated in Figure 62 where the effective thermal conductivity, due to radiation, of a pore having a thickness dp, is shown as a function of temperature. The thermal conductivity of solid ZrO_2 is shown also, for comparison. It is interesting to note that a pore having a thickness on the same order of magnitude as the coating thicknesses being investigated in this study (0.10 mm), even at high temperatures (1600°C), has an effective thermal conductivity of only 3 percent that of solid ZrO_2 . This agrees with the previous conclusion that radiation across the individual pores makes a negligible contribution to the overall heat flux through a porous coating.

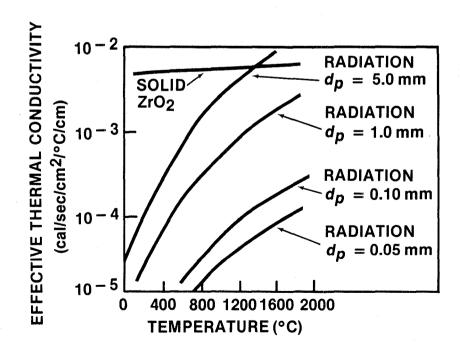


Figure 62. Radiation

The nature of the interface between two phases in a mixure may also affect the properties of the mixture. In porous ceramic coatings this effect may influence the resultant thermal conductivity, since heat transfer across the interface must occur by natural convection.

Other factors that have been considered in their effect on the properties of two-phase mixtures having one dispersed phase are the maximum packing fraction and the shape, agglomeration, orientation and distribution of the dispersed particles. The packing fraction accounts for the fact that the dispersed phase cannot pack in such a manner as to completely fill a space. Nielsen (Ref. 7) has described an analytical approach for calculating the properties of mixtures which takes into account these latter factors. The general mixture rule proposed by Nielsen is:

$$\frac{P_{C}}{P_{m}} = \frac{1 + A B v_{d}}{1 - B \psi_{v_{d}}}$$
 (19)

where

$$A = 1/(K_E - 1)$$
 (20)

$$B = \frac{P_{c}/P_{d} - 1}{P_{c}/P_{d} + A}$$
 (21)

$$= 1 + \left(\frac{1 - m}{\phi_m^2}\right) \phi_d \tag{22}$$

and P_c = property of continuous phase P_d = property of dispersed phase P_m = property of mixture

= Einstein coefficient

= volume fraction of dispersed phase

maximum packing fraction

The constant A depends upon the shape of the dispersed particles, their state of agglomeration, their orientation and the nature of the interface. "A" can vary from zero to infinity and is related to the Einstein coefficient which can be calculated theoretically or derived empirically. The factor ψ takes into account the maximum packing fraction which is defined as:

$$\phi_{\rm m} = \frac{\text{true particle volume}}{\text{volume occupied by particles}}$$

Values of m can be estimated or measured in a variety of ways.

Table 12 presents some values of $\phi_{\rm m}$ which were derived largely from theory. Table 13 presents some values of KF derived for mechanical properties of mixtures. In general, values of K_E are usually slightly greater for thermal properties than for mechanical properties, but often only small errors will result from using the mechanical values for thermal calculations.

Table 12 $\mbox{Values of $\phi_{\rm m}$ for Various Mixtures }$

-		-
Particles	Type of Packing	$\phi_{ m m}$
Spheres Spheres Spheres Spheres Spheres Spheres	Hexagonal close packing Face centered cubic Body centered cubic Simple cubic Random close packing Random loose packing	0.7405 0.7405 0.60 0.5236 0.637 0.601
Fibers Fibers Fibers	Parallel hexagonal Parallel cubic Parallel random	0.907 0.785 0.82
Cubes	Random	0.70
Rods	L/D = 4, random three dimensional (approx.)	0.625
Rods	L/D = 8, random three dimensional (approx.)	0.48
Rods	L/D = 16, random three dimensional (approx.)	0.30
Rods	L/D = 40, random three dimensional (approx.)	0.13
Rods	L/D = 70, random three dimensional (approx.)	0.065

The original Einstein equation, derived for the viscosity of suspensions of rigid spheres, can be generally expressed as:

$$P_{m}/P_{c} = 1 + K_{E} v_{d}$$
 (23)

for low concentrations of the dispersed phase when P_d/P_c is very large. This form of the equation may be applied to porous coatings if P is used to represent thermal resistivity, since the resistivity of air is much greater than that of the ceramic. This expression provides a simple method of determining K_E from experimental data. Equation (23) can thus be transformed to:

$$k_{\rm m}/k_{\rm C} = 1/(1 + K_{\rm E} v_{\rm d})$$
 (24)

for low values of v_d and large values of k_c/k_d . This is referred to here as the simplified Einstein equation.

Table 13 $Values of K_F$ for Various Mixtures

		
Type of Dispersed Phase	Orientation of Particles and Type of Stress	K _E
Dispersed spheres	Any. No slippage	2.50
Dispersed spheres	Any. Slippage	1.0
Spherical aggregates of spheres	Any. $\phi_{\rm a}=\phi_{\rm m}$ of spheres within aggregate	2.5/ a
Cubes	Random (approximate)	3.1
Uniaxially oriented fibers	Fibers parallel to tensile stress component	2L/D
Uniaxially oriented fibers	Fibers perpendicular to tensile stress component	1.50
Uniaxially oriented fibers	Longitudinal-transverse shear	2.0
Uniaxially oriented fibers	Traverse-transverse shear	1.5
Uniaxially oriented fibers	Bulk modulus	1.0
Fibers randomly oriented in three dimension	Shear. (Approximate)	L/2D

Comparison of Theoretical Conductivity Calculations and Experimental Data

Three methods of calculating the theoretical thermal conductivity of a porous coating have been suggested: (1) the Maxwell theory, (2) the Einstein equation and (3) the simplified Einstein equation. Each of these methods was applied to a porous zirconium oxide coating to predict effective thermal conductivity as a function of porosity. Thermal conductivity values from the literature were used for solid Zr0₂8Y₂0₃ and air, both at 1200°K. The results of these analyses are presented in Figure 63 and are compared to the experimental data presented earlier.

Theoretical curve A in Figure 63 was calculated from equation (16) based on Maxwell's mixture theory. This equation expresses the effective thermal

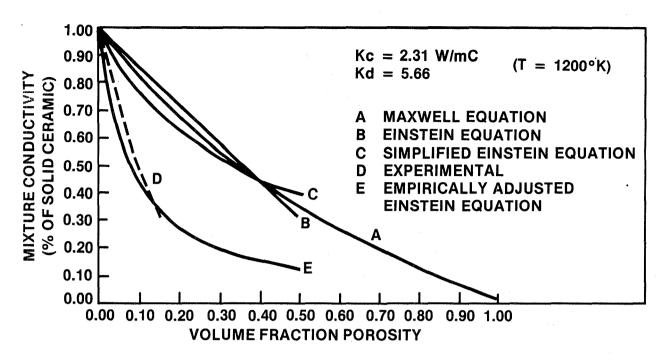


Figure 63. Theoretical Thermal Conductivity of Porous Coating

conductivity of the mixture in terms of the conductivities and volume fractions of the individual phases. It does not account for pore size, distribution, agglomeration, orientation, or interface effects.

Theoretical curve B was established using equation (19) (the Einstein equation). It was necessary to estimate values of K_E and ϕ_m to apply this equation. A value of K_E = 3.5 was selected based on the value of K_E for random cubes presented in Table 13. The selected value of K_E is slightly greater than the value in the table for mechanical properties, to compensate for the difference between K_E values for mechanical and thermal properties. A value of ϕ_m = 0.70 was selected based on the value for random cubes presented in Table 12.

The properties, P, in equation (19) represent thermal conductivities for this analysis. This equation presumably accounts for the distribution, agglomeration, orientation and interface effects of the dispersed pores. It is recognized that the assumed values of K_E and ϕ_m are educated guesses and should be investigated in more detail if this analytical approach is pursued further.

Theoretical curve C was established using equation (24) (the simplified Einstein equation). Only a value for K_E was needed for this analysis. Again, this value was selected as $K_F = 3.5$.

Curve D in Figure 63 represents the experimental data generated earlier. The value of solid ceramic conductivity used to generate this curve is a commonly quoted literature value, $k_C = 2.06 \text{ W/mC}$. It should be noted that besides the possible error in the value used for k_C (+ 9% based on reported data) this

experimental data curve also represents two measured values, k_{m} and P_{t} , which are subject to experimental error.

It is obvious from Figure 63 that the experimental data does not agree well with the theoretical analyses. The three theoretical models all agree fairly well with each other in the low porosity region; the largest discrepancy (about 15% difference) being between the two versions of the Einstein equation. The Maxwell conductivity curve lies midway between these two.

The fifth curve in Figure 63 represents an empirically adjusted Einstein equation. In this case two values of $K_{\mbox{\scriptsize F}}$ were calculated from the experimental data using equation (24). These values are:

 $K_E = 11.74 @ 10\% \text{ porosity}$ $K_E = 16.32 @ 15\% \text{ porosity}$ $K_E = 14 \text{ average}$

The average value of K_F was used in equation (24) to construct curve E, which matches the experimental data reasonably well (within 9.5%). Although this value of K_{E} is larger than values quoted in the literature, it is acceptable since there is no theoretical upper limit to the value of K_{E} . The equation for curve E is:

$$k_{\rm m}/k_{\rm c} = 1/(1 + 14 v_{\rm d})$$
 (25)

Equation (19) (the Einstein equation) was also used to generate an empirically calibrated expression for effective thermal conductivity. In this case a value of $\phi_{\rm m}$ = 0.70 was assumed and values of K_E were calculated using the experimental thermal conductivity data in equations (19) through (22). This analysis yielded an average value of K_E = 1.06, which was used with the assumed value of ϕ_{m} to calculate a curve of thermal conductivity vs. porosity. The resulting curve fell almost exactly on the empirically adjusted curve created previously using the simplified Einstein equation.

No attempt to empirically calibrate equation (18) was made. This expression has no inherent variables which can be manipulated. It appears from Figure 63 that the difference between this equation (curve A) and the experimenal data is more than just a constant, and therefore, would probably involve an exponential factor of some form.

Overall, equation (25) appears to provide the simplest, reasonably accurate expression for calculating the effective thermal conductivities of the ceramic coatings in this study. It is recognized that the experimental data base should be expanded to provide further verification of this relationship. It is likely that a more accurate value of K_{F} may be determined through additional testing, but that the form of equation (25) will still apply.

3.3.3 Discussion of Model Development

A practical method for analytically modelling TBC's has been developed. This method basically consists of two parts. First, the material properties of the TBC are determined, either experimentally or analytically. Second, these material properties are used in a finite element model of the coating system to predict the thermo-mechanical response of the coating to specific operating conditions. This model can also be used to predict the effects of coating process parameters on coating performance. This requires that the effect of the process parameters on the coating properties first be determined. Then, these material property changes can be incorporated into the finite element model, to evaluate the effects on coating performance.

The material properties used in this study were either measured or taken from the literature. A more complete analytical model could be constructed if the material properties were analytically derived. This could be accomplished by establishing empirical relationships between coating properties and coating deposition parameters. Coating properties could then be predicted by monitoring the coating deposition parameters. An alternative approach might be to relate all coating properties to easily measured coating characteristics, such as porosity.

The objective of the porous coating model was to establish an analytical method of predicting the thermal conductivity of a coating based on porosity. Thermal conductivity was selected for this investigation since it is one of the most critical properties in these analyses. It was intended to uniquely define the porous structure of a coating in terms of geometric parameters, and relate these parameters to thermal conductivity. These relationships would then be empirically adjusted. A real coating could then be defined in terms of these geometric parameters and it's thermal conductivity calculated.

The porous coating model was intended to be developed one step further. A relationship between coating deposition parameters and the geometric coating parameters was to be established empirically. This model could then be used to predict the thermal conductivity of a coating based on its deposition parameters.

However, the porous coating model was not sufficiently developed to achieve these results. The geometric parameters selected to characterize a porous coating structure proved inadequate. Despite the obvious inconsistencies in the model, however, much of the analysis was consistent and produced logical results. It is believed that the basic model concept is sound, but requires further development. The geometric coating parameters need to be re-evaluated and perhaps redefined or expanded.

The theoretical models for coating thermal conductivity appear to have merit. Empirical adjustment of the Einstein equation produced a reasonably accurate expression for thermal conductivity in terms of porosity. However, additional empirical data is required to fully evaluate these models.

3.4 COMPUTER ANALYSES

3.4.1 Coating Strength Data Analyses

Coating strength data were generated using the four-point flexure test, for the three coating types previously described: those applied at the 20, 30 and 40 kW power levels. These are referred to as coating types A, B and C, respectively. Two sets of coating strength data were generated at different points in time. Each set tested three thicknesses of each coating type. Set number 1 tested coatings of thicknesses 0.0508, 0.1016 and 0.1524 mm; set number 2 tested coatings of thicknesses 0.0254, 0.0508 and 0.1016 mm. Three specimens were tested for each coating type and thickness combination. The strength data is presented in Table 14a and 14b, along with the measured loads and deflections at failure and calculated radii of the neutral axes. Coating strength is defined here as the maximum compressive strain in the coating at the time of failure ($\epsilon_{\rm m}$). Also reported is the strain at the bond interface of the coating ($\epsilon_{\rm b}$), and at the coating mid-thickness ($\epsilon_{\rm c}$).

The coating strength data is plotted in Figure 64 as maximum coating strain vs. coating thickness. Although coating thickness was not known to affect coating strength, it did provide a convenient graphical means for comparison of coating strengths. Curves A, B and C represent the original data set and curves D, E and F represent the second data set. Each data point represents the average of three strength tests.

It is difficult to make intuitive conclusions about the relative strengths of the three coating types, based on the data presented in Figure 64. The strengths of each coating type overlap each other, within each data set, and exhibit inconsistent trends as functions of thickness. It is apparent, however, that the strengths of the coatings in the second set were significantly higher than those in the first. The reason for this has not been determined. Although the coating application parameters were the same for both coating specimen sets, there may have been a subtle improvement in the application process for set #2 due to the experience of coating set #1.

The apparently inconsistent results from the coating strength tests are not unreasonable. Rarely do ceramic materials of any type exhibit deterministic strengths (Ref. 8). It is often convenient to employ statistics in evaluating the strength of ceramic materials. Thus, statistical analyses of the coating strength data were used to compare the strengths of the three coating types.

Because of the obvious difference in strengths between the two sets of coating tests, each set was evaluated individually. A matched-pair t test (Ref. 9) was used to compare the strengths of the three coatings in each set. This statistical method is used to make inference about the mean of the difference between two matched groups. In this case the strengths of coatings having the same thickness were matched to form pairs of data. This minimized the possible influence of thickness on the strength comparison.

Table 14a
Four-Point Bend Specimen Results - Data Set 1

			·	Г					r
		t	P	Y	R1	€b	€ .	ϵ_{m}	R
		"	-	•	Inner	Interface	Coating	Surface	Neutral
	Coating	Thickness	Load	Deflection	Radius	Strain	Strain	Strain	Radius
Number	Type	(mm)	(kg)	(mm)	(mm)	x 10 ⁻²	x 10 ⁻²	x 10 ⁻²	(mm)
	-715	((11-9)	(/	(,				,,
		0.0508	15.5	5.18	33.27	-0.801	-0.877	-0.952	33.59
1	A	0.0508	16.0	5.59	31.24	-0.840	-0.921	-1.001	31.56
2				1		į.		ı	ł
3)		16.5	6.10	28.96	-0.889	-0.976	-1.063	38.66
4		0.1016	21.0	6.58	27.18	-0.703	-0.888	-1.073	27.47
5	ļ	1	22.5	6.58	27.18	-0.703	-0.888	-1.073	27.47
6			20.5	6.38	27.94	-0.691	-0.871	-1.051	28.24
7		0.1524	27.0	8.18	23.11	-0.575	-0.900	-1.226	23.40
8)		28.7	8.18	23.11	-0.575	-0.900	-1.226	23.40
9			27.5	7.19	25.40	-0.550	-0.847	-1.143	25.69
10	13	0.0508	17.0	5.79	30.23	-0.861	-0.944	-1.027	30.54
11			18.0	5.99	29.46	-0.870	-0.963	-1.048	29.78
12			18.0	5.79	30.23	-0.861	-0.944	-1.143	25.69
13	'	0.1016	29.0	7.39	24.89	-0.742	-0.943	-1.145	25.18
14			24.0	8.18	23.11	-0.775	-0.992	-1.209	23.40
15			28.7	7.87	23.62	-0.765	-0.978	-1.190	23.91
16		0.1524	26.8	7.19	25.40	-0.550	-0.847	-1.143	25.69
17			25.0	6.20	28.70	-0.519	-0.782	-1.044	29.00
18			27.1	7.77	23.88	-0.566	-0.882	-1.197	24.17
19	С	0.0508	8.3	8.69	22.10	-1.082	-1.196	-1.309	22.39
20			10.6	6.99	25.91	-0.965	-1.062	-1.159	26.21
21	[
22		0.1016	23.0	8.99	21.59	-0.807	-1.039	-1.271	21.87
23			22.5	6.58	27.18	-0.703	-0.888	-1.073	27.47
24		ĺ	21.5	7.29	25.15	-0.737	-0.937	-1.137	25.44
25		0.1524	35.0	7.49	24.64	-0.558	-0.864	-1.169	24.93
26			22.0	11.18	18.80	-0.627	-1.027	-1.427	19.07
27			21.0	8.38	22.61	-0.580	-0.913	-1.246	22.89

Table 14b

Four-Point Bend Specimen Results - Data Set 2

Number	Coating Type	t Thickness (mm)	p Load (kg)	Y Deflection (mm)	R ₁ Inner Radius (mm)	6 b Interface Strain x 10 ⁻²	€ _C Coating Strain x 10 ⁻²	€ _m Surface Strain x 10 ⁻²	R Neutral Radius (mm)
1	A	0.0254	15.3	8.13	23.18	1.252	1.306	1.360	23.50
2			14.6	7.87	23.73	1.229	1.281	1.334	24.05
3			13.8	8.13	23.18	1.252	1.306	1.360	23.50
4		0.0508	13.7	7.62	24.32	1.204	1.256	1.308	24.64
5	\	'	13.8	7.37	24.95	1.180	1.230	1.280	25.27
6			13.7	6.86	26.38	1.127	1.175	1.222	26.70
7		0.1016	14.8	7.62	24.32	1.204	1.256	1.308	24.64
8			17.3	8.13	23.18	1.252	1.306	1.360	23.50
9			16.0	7.37	24.95	1.180	1.230	1.280	25.27
10	В	0.0254	14.0	8.13	23.18	1.046	1.154	1.262	23.47
11			13.6	8.13	23.18	1.046	1.154	1.262	23.47
12	(1							l
13		0.0508	17.5	8.38	22.67	1.063	1.173	1.284	22.96
14			17.1	8.13	23.18	1.046	1.154	1.262	23.47
15	1	ĺ	17.7	8.13	23.18	1.046	1.154	1.262	23.47
16		0.1016	20.0	10.67	19.31	1.190	1.319	1.449	19.60
17			20.1	10.67	19.31	1.190	1.319	1.449	19.60
18			19.8	9.65	20.58	1.138	1.260	1.382	20.85
19	С	0.0254	20.0	9.91	20.23	0.837	1.085	1.333	20.50
20	1	1	19.5	9.14	21.33	0.812	1.047	1.282	21.61
21			17.0	8.64	22.19	0.794	1.020	1.246	22.48
22		0.0508	22.2	9.65	20.58	0.829	1.073	1.317	20.85
23			22.0	7.37	24.95	0.741	0.942	1.143	25.25
24			21.0	9.65	20.58	0.829	1.073	1.317	20.85
.25		0.1016	23.3	9.40	20.94	0.821	1.060	1.300	21.21
26			19.3	6.86	26.38	0.716	0.907	1.097	26.67
27		, '	20.7	6.86	26.38	0.716	0.907	1.097	26.67

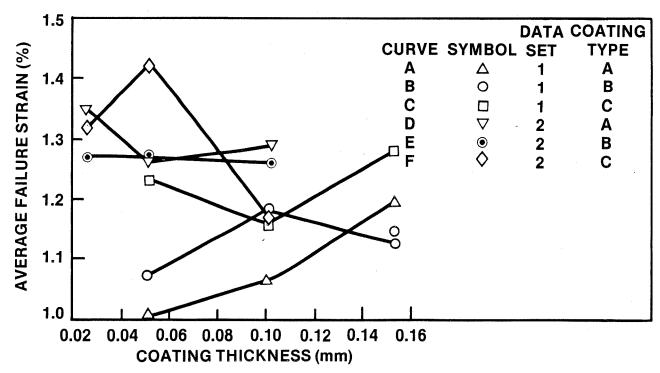


Figure 64. Coating strength Data From Four-Point Flexure Tests

The Student's t distributions may be used for inference about the mean of a sample group when (1) the population distribution is normal (or at least symmetrical and unimodal), (2) the population variance is unknown and estimated by the sample variance and (3) the sample is random. The latter two conditions are inherently met by the nature of the coating strength tests. However, the strengths of ceramics do not necessarily exhibit a normal distribution. Thus, condition (1) may not be satisfied. To determine how well the coating strength data generated in this study matched a normal distribution, a Chi-squared goodness-of-fit test (Ref. 9) was employed. A 95 percent confidence level was selected to evaluate the hypothesis that the coating strength samples were from a normal distribution. These tests indicated that the hypothesis could not be rejected for data set 2 and could be rejected for data set 1. The calculated value of Chi-squared for data set 2 was actually quite close to the rejection limit. Thus, little evidence of a normal distribution was detected through these analyses.

Despite this result, condition (1) may be reasonably satisfied by applying the central limit theorem. This theorem states that even if a population is not normal, the sampling distribution of averages for that population is approximately normal for a large number of samples (>30). The sampling distribution of averages is the probability distribution associated with the sample average (y). This distribution consists of all possible values of y, for a fixed sample size, and the probabilities associated with these values of the random variable.

In the case of the coating strength data generated for this program, only nine strength values per coating type, per data set, were obtained. Although

this does not meet the minimum sample size recommended to approximate a normal distribution, it was used for this analysis and this factor should be considered when evaluating the results. The three coating strengths measured for each coating thickness/type combination were averaged to create the data base for this analysis.

The null hypothesis used in these analyses was $H_0:\mu_d=0$ and the alternative hypothesis was $H_a:\mu_d\neq 0$ in which μ_d is the population mean for the difference in strength between two coatings. This hypothesis was tested to compare pairs of coating strength data; three tests were required for each data set: A vs. B, A vs. C and B vs. C. The Student's t values were calculated as follows:

$$S_d^2 = \frac{y_d/n}{\epsilon y_d^2 - (\epsilon y_d)^2/n}$$

$$t = \frac{\bar{y}_d - \mu_{do}}{S_d / \sqrt{n}}$$

where

n = number of pairs

 y_d = strength difference y_d = average strength difference y_d = sample variance

 $\mu_{ extsf{do}}$ = mean strength difference assumed by the null hypothesis

A confidence limit of 95 percent was used for these analyses. Thus, $\alpha = 0.05$ where α corresponds to the probability that t exceeds a corresponding value in the Student's t tables. The degree of freedom ($\gamma=n-1$) for these tests was always 2. The t values are therefore t_{0.025}, $\gamma=0.025$, $\gamma=0.$ percent, assuming the null hypothesis is correct.

The coating strength data used in these analyses are shown in Table 15. The t statistics, and some of the intermediate calculated values, for these data are presented in Table 16.

Inspection of the calculated t statistics for data set 1 suggests the following:

- The strength of coating A is not significantly different from the strength of coating B.
- The strength of coating A is significantly different from the strength of coating C.
- The strength of coating B is not significantly different from the strength of coating C.

 $\label{total} \mbox{Table 15}$ Coating Strength Data for Statistical t-Test Analysis

	ta		Average Comp	ressive Failure	e Strain (%)
Se Num	t ber	Thickness (mm)	Coating A	Coating B	Coating C
1		0.0508 0.1016 0.1524	1.005 1.066 1.198	1.073 1.181 1.128	1.234 1.160 1.281
2)	0.0254 0.0508 0.1016	1.350 1.260 1.290	1.270 1.270 1.260	1.320 1.430 1.170

Table 16a

Matched Pair t-Test Calculations - Data Set 1

	A vs. B		A	A vs. C		vs. C
Thickness	Difference ^Y d1	(Difference) ² yd1	Difference Yd2	(Difference) ² y _{d2}	Difference Y _{d3}	(Difference) ² Yd3
0.0508 0.1016 0.1524	-0.068 -0.115 0.07	0.004624 0.013225 0.0049	-0.229 -0.094 -0.083	0.052441 0.008836 0.006889	-0.161 0.021 -0.153	0.025921 0.000441 0.023409
Σ	-0.113	0.022749	-0.406	0.068166	-0.293	0.049771
Average difference \overline{y}_d	-0.	.0377	-0.1356		-0.09767	
(Sample variance) ² S _d ²	0 .	.007779	0.00661		0.01058	
test statistic t	2.2	219	-8	. 649	~1 .	. 645

Table 16b

Matched Pair t-Test Calculations - Data Set 2

	7 A	/s. B	Α .	vs. C	B vs. C	
Thickness (mm)	Difference ^Y d1	(Difference) ² y _{d1}	Difference Y _{d2}	(Difference) ² y _{d2}	Difference Y _{d3}	(Difference ²)
0.0254 0.0508 0.1016	0.08 -0.01 0.03	0.0064 0.0001 0.0009	0.03 -0.17 0.12	0.0009 0.0289 0.0144	-0.05 -0.16 0.09	0.0025 0.0256 0.0081
Σ	0.10	0.0074	-0.02	0.0442	-0.12	0.0362
Average difference \bar{y}_d	0.03	3333	-0.00667		-0.04	
(Sample variance) ² Sd	0.00	0203	0.02203		0.00577	
test statistic t	1.28	31	-0.0	778	-0.	•912

These statistical conclusions can be interpreted to indicate that there is a strength difference among the three coating types. Since A and C are probably different, and B cannot be distinguished from either A or C, it can be assumed that B has a strength somewhere between A and C. Thus, these results suggest that there was an increase in coating strength associated with an increase in plasma spray power level for the coatings in data set 1.

Assuming that the conclusions made from the analysis of data set 1 are valid, a plot of average failure strain vs. power level was constructed; see Figure 65. Table 16 shows the pertinent data for this curve. This figure serves to illustrate the results of the data set 1 strength analysis and is not intended to be taken conclusively.

Inspection of the calculated t statistics for data set 2 suggest the following:

- . The strength of coating A \underline{is} not significantly different from the strength of coating B or C.
- . The strength of coating B \underline{is} not significantly different from the strength of coating A or C.

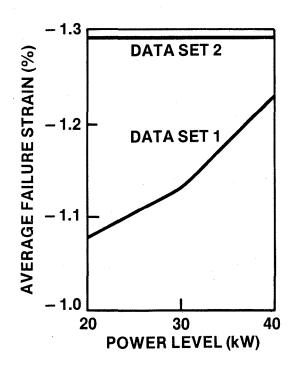


Figure 65.

Average Coating Failure Strain Versus Plasma Spray Power Level

These statistical conclusions indicate that there was no difference in strength among the three coating types in data set 2. This analysis suggests, therefore, that there was no effect of power level on the strengths of the coatings in data set 2. The average of all coating failure strains in data set 2 was calculated as -1.29 percent. This result was added to the data set 1 results in Table 17 and Figure 65.

The results of the strength data analyses for data sets 1 and 2 are basically inconsistent. Data set 1 indicates an effect of power level on coating strength whereas data set 2 does not. In addition, data set 2 shows a significantly greater coating strength than data set 1. These inconsistencies suggest that additional testing is required before any conclusions can be made about the true strength of these coatings or the effect of power level on coating strength.

If we assume that power level does not have an effect on coating strength, then each data set represents a characteristic strength distribution for the coating system tested. Although the difference in strength between each data set is obvious, it is currently unexplained. Thus, although each data set theoretically represents the same coating system, they will be evaluated separately, without assuming that either one is incorrect.

The strength data from data sets 1 and 2 were used to evaluate the probability of coating survival vs. coating strain. The coating strength data from each set was ranked according to value. The probability of survival (P_S) for each data point was then calculated from:

$$P_S = 1 - R/(N + 1)$$
 (26)

Table 17

Average Coating Strength as a Function of Plasma Spray Power Level

Data Set	Coating Type	Power Level (kW)	Average Failure Strain (%)	Range
1	A	20	-1.09	-0.952 to -1.226
	В	30	-1.13	-1.027 to -1.209
	С	40	-1.23	-1.073 to -1.427
2	A,B,C	20,30,40	-1.29	-1.170 to -1.430

where R is the rank of the strength value and N is the number of specimens in each set. This data is shown plotted in Figure 66. The difference in strengths between the two data sets is illustrated clearly in this figure.

Computerized regression analyses were used to fit fourth order polynominal equations to this data. These resulting equations are:

data set 1:
$$(r = 0.991)$$

$$P_s = -83.2 + 272.1e - 319.0 e^2 + 160.4e^3 - 29.3e^4$$
 (27)

data set 2: (r = 0.989)

$$P_s = 693.1 - 2331.1e + 2926.9e^2 - 1622.4e^3 + 334.7e^4$$
 (28)

where e is the strain in percent, and r is the correlation coefficient.

Other probability distributions were also evaluated for their applicability to the coating strength data. For example, the Weibull relationship (Ref. 8), which is often found applicable to ceramic strength data, was investigated. The Weibull relationship is:

$$\ln \ln [1/(1 - F)] = m \ln (e/e_0)$$
 (29)

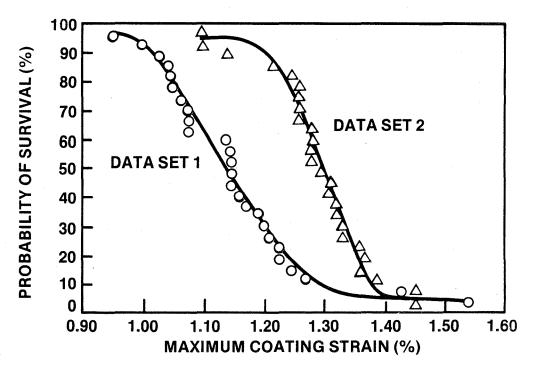


Figure 66. Survival Probablility Versus Coating Strain

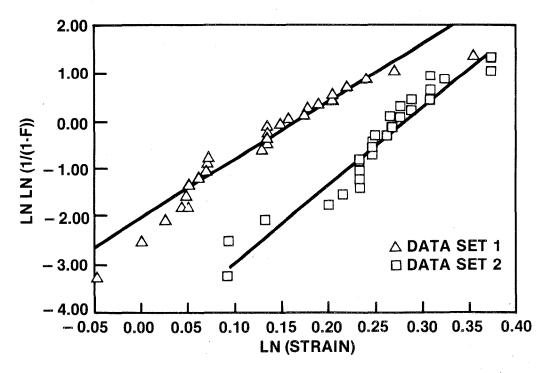


Figure 67. Coating Strength Weibull Plots

data set 1
$$(r = 0.965 m = 12)$$

$$\ln \ln (1/(1-F)) = 12.12 \ln (e) - 2.075$$
 (30)

data set 2 (r = 0.970 m = 16)

$$\ln \ln \left(1 / (1 - F) \right) = 16.16 \ln \left(e \right) - 4.577$$
 (31)

where r is the correlation coefficient, and m is the slope of the curve (or the Weibull modulus).

Inspection of Figure 67 shows the marked difference in strength observed between data sets 1 and 2. There is also a slight difference in the slopes of the two curves. The slopes of these curves are referred to as the Weibull moduli ["m" in equation (29)]. A large value of m indicates a small spread in strength values. Therefore, Figure 67 indicates that data set 2 was not only stronger but also had a narrower strength distribution. This again may be attributed to subtle improvements in the coating application process between the two specimen sets.

The Weibull probability of failure plots have quite acceptable correlation coefficients. Also, the Weibull moduli calculated from this data are in the range of values quoted in the literature for many monolithic ceramics (m = 4 to 20). The significance of this has not been determined.

Either of the probability of failure distributions presented for coating strain can be used (graphically or mathematically) to predict coating reliability under specific operating conditions. Alternatively, the limiting operating conditions for a coating can be established for a desired survival probability. These analyses first require a thermomechanical model of the coating system, to calculate the strain in the coating. This strain can then be used to predict the probability of coating survival, using the probability distributions just presented.

3.4.2 Performance Analyses of Actual TBCs

Performance predictions for the thermal barrier coatings produced in this study were made using the thrust chamber model described earlier in this report. A range of steady state operating conditions was analyzed which included a variety of heat fluxes, temperature gradients and coating thicknesses. Measured material properties of the coatings were used; each coating type was unique in its value of thermal conductivity. An average thermal conductivity was used in some analyses, to predict the effects of other parameters.

Material properties other than the thermal conductivity of the coating were obtained from the literature. These properties are listed in Table 1. Since thermal conductivity is the only coating property of consequence for steady state thermal analyses, the thermal analyses of these coatings are expected to be quite accurate.

The structural analyses depend upon values for the coefficient of thermal expansion, Poisson's ratio and the elastic modulus. Although literature values were used for these properties, little discrepancy between the structural analytical results and the true behavior of these coatings is expected. The coefficients of thermal expansion measured in this study agreed well with the literature values, thus giving confidence in the use of the literature values in the analyses. The literature values were used because they were defined as functions of temperature whereas the measured values were obtained at a specific temperature. Furthermore, the use of the elastic modulus was often eliminated by evaluating coating strength in terms of strain. Thus, Poisson's ratio, which is of secondary importance, remains as the only material property of any uncertainty in the structural analyses. Future studies may focus on obtaining accurate measurements of this property, as well as better measurements of the other properties, to improve the overall coating model.

Three values of thermal conductivity were used for the coatings in these analyses: 0.60, 0.935 and 1.0 W/mC. These represent the highest and lowest measured values of the three coating types tested, and an average of the 30 and 40 kW coatings. This average was taken from the most consistent experimental data that was obtained.

The cold surface temperature for these analyses was 200°C , unless otherwise stated. This temperature was thought to be most representative of a steady state rocket engine operation. Some analyses were performed with a -20°C cold surface temperature.

Parametric coating systems analyses were conducted using four specific heat fluxes: 16, 49, 82 and 164 W/mm². These heat fluxes cover the range of values typically encountered in uncoated copper rocket thrust chamber walls, the highest value representing an extreme condition at the throat section. Applying a TBC will implement a reduction in heat flux through the wall, by increasing the operating wall surface temperature and thus decreasing the temperature difference between the gas and the wall.

A simple analysis indicates that this reduction in heat flux may be significant. For example, let us assume that the heat flux to the hot surface of the thrust chamber is due only to convection of the hot gases, and that the heat transfer coefficient and surface area are constant. Then, any change in the temperature difference between the gas and the hot surface will result in a proportional change in heat flux.

Let us assume that the hot gas temperature is 3000°C . Data from NASA indicates that the heat flux through a bare copper thrust chamber wall may reach $164~\text{W/mm}^2$. Let us further assume that the surface of the copper wall reaches its melting point of 1100°C at this condition (extreme case). We can now calculate the expected heat flux through a coated thrust chamber wall operating with an acceptable surface temperature of 2000°C from

$$Q_c = Q_b (T_g - T_s)_c / (T_g - T_s)_b$$
 (32)

where 0 = heat flux

 T_g = gas temperature T_S = surface temperature

and the subscripts c and b refer to the coated and bare thrust chamber walls, respectively. This calculation indicates that the heat flux through the coated wall may be only 53 percent of the heat flux through an uncoated wall, or 86 W/mm^2 at the maximum condition.

Based on this calculation, it is assumed that the heat flux value of 82 W/mm^2 used in these analyses is representative of the maximum actual heat flux in the throat section of a coated high pressure rocket thrust chamber. The lower heat flux values are expected to represent the non-throat sections of the coated wall, or part-load operating conditions. The analyses using a heat flux of 164 W/mm^2 are useful in establishing trends from the parametric study results.

Figure 68 shows the temperature profiles through a coating system having a 0.0254 mm TBC with average thermal conductivity, for two heat fluxes: 49 and 82 W/mm². These are medium and high heat flux values for a coated high pressure rocket thrust chamber. It is obvious that most of the temperature drop occurs across the TBC, subjecting the bond and substrate to a relatively small thermal gradient, as intended. As also expected, the thermal gradient through the coating is greater, and the surface temperature much higher, with the higher heat flux. The lower heat flux requires a surface temperature approximately equal to the maximum desirable ceramic temperature (~1900°C).

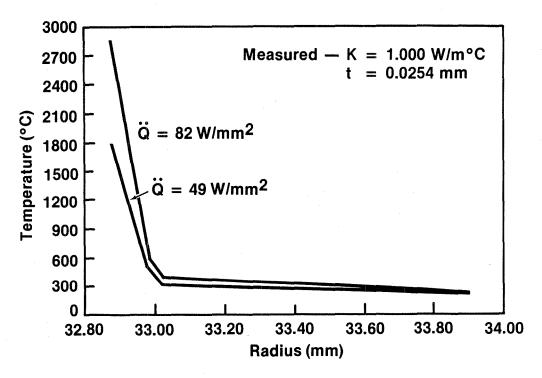


Figure 68. Predicted Coating Temperature Profile

Figure 69 shows the hoop (or axial) stress profile that results in this coating system from the imposed thermal gradients. These curves show the TBC and bond coat in compression and the substrate primarily in tension. The higher heat flux induces higher compressive stresses in the ceramic and higher tensile stresses in the substrate, than the lower heat flux. These differences are approximately proportional to the difference in temperature drop between the two cases. The magnitudes of the coating stresses are relatively high, on the order of 500 MPa. The significance of these stresses will be discussed later.

Figure 70 shows the radial stress profile through the same coating system. The magnitude of these stresses are quite small, (0-3 MPa), but are greatest in the copper substrate. Also, the effect of heat flux is most significant in the substrate, whereas the temperatures, hoop and axial stresses were most affected in the coating.

Figure 71 represents the temperature profiles through two coating systems having a 0.0254~mm TBC with different thermal conductivities: 0.6~and~1.0~W/mC. The heat flux through these coatings is $82~\text{W/mm}^2$. The thermal gradient and surface temperature of the ceramic coating are much greater with the lower value of k. No difference was calculated in the thermal gradients of the bond and substrate between the two cases, and again, this thermal gradient is relatively small. In this case, both of the ceramic surface temperatures exceed the maximum desirable ceramic operating temperature.

Figure 72 shows the hoop (or axial) stress profiles through the same coating system. Again, the TBC is shown under significant compressive stress (500-800

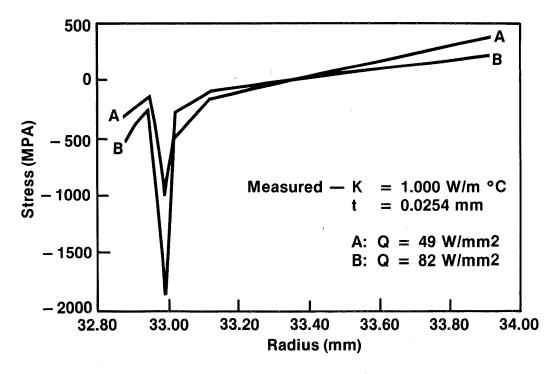


Figure 69. Predicted Hoop and Axial Coating Stress Profile

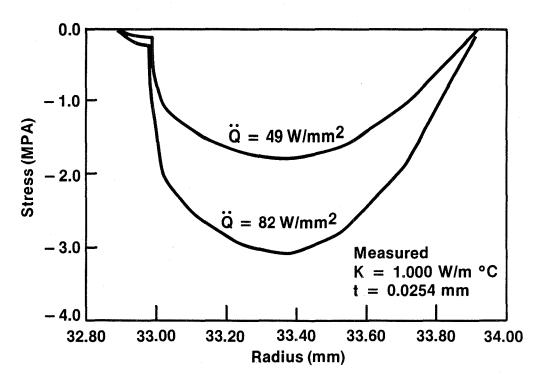


Figure 70. Predicted Radial Stress Profile Through Coating Using Average Measured Thermal Conductivity

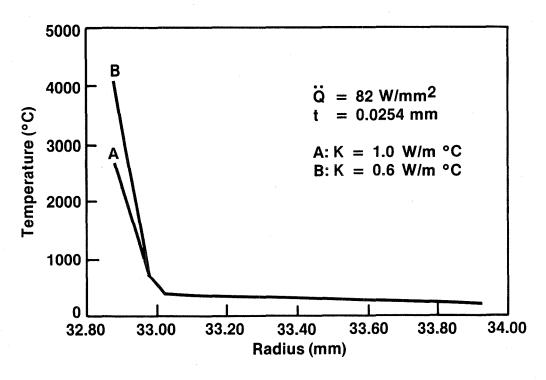


Figure 71. Coating Temperature Profile

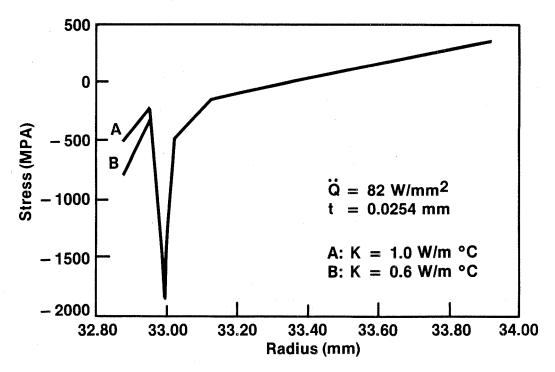


Figure 72. Hoop and Axial Coating Stress

MPa). The bond and some of the substrate are also in compression, while most of the substrate is in tension. The coating with higher thermal conductivity is seen to incur significantly less stress than the other coating, while the remainder of the coating systems stresses are identical.

These initial analyses indicated that a coating thickness of 0.0254 mm may be inadequate for very high heat flux conditions. Thus, a series of analyses were conducted to determine the effect of coating thickness on coating surface temperature at the maximum heat flux condition expected in a typical high pressure rocket thrust chamber: 82 W/mm². The results of these analyses are presented in Figure 73, for two values of coating thermal conductivity: 0.6 and 1.0 W/mC. A desirable maximum ceramic operating temperature is shown here as 1900°C.

Inspection of Figure 73 is very informative. This analysis demonstrates the importance of accurately knowing the thermal conductivity of a TBC and of the ability to accurately, and precisely, control coating thickness. For example, suppose that a 0.015 mm coating with $k=1.0\,$ W/mC is specified for this high heat flux application, to operate at the limiting surface temperature. If the conductivity of the coating is actually 0.6 W/mC, the ceramic surface temperature must be increased to approximately 2600°C to accommodate the same heat flux.

Similarly, assume that a 0.009 mm coating with k=0.6 W/mC is specified for this same application, again to operate at the limiting surface temperature. If a portion of this coating turns out to be 0.015 mm thick, the ceramic temperature may exceed the desired limit in that area if the heat flux is

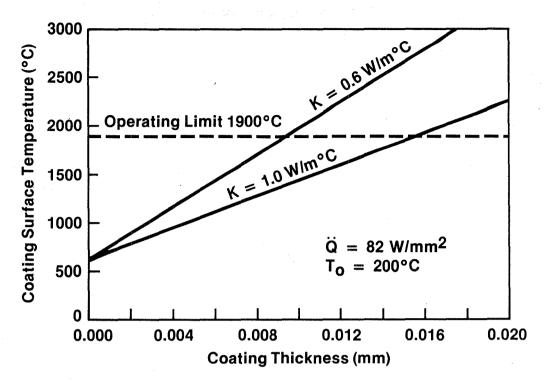


Figure 73. Effect of Coating Thickness on Surface Temperature for Fixed Heat Flux

maintained. The effect of thickness is seen to be more pronounced for the lower thermal conductivity coating, as its slope in Figure 73 is steeper than the other.

Figure 74 shows the effect of coating thickness on the maximum compressive coating strain in the TBC, for two coating thermal conductivities (0.6 and 1.0~W/mC) and a heat flux of 82 W/mm². Coating strain is seen to increase with coating thickness, and at a faster rate for the lower thermal conductivity coating. Again, the significance of thermal conductivity is demonstrated in this figure. For example, a 0.010 mm coating with k = 1.0~W/mC operating at this condition would experience a compressive strain of approximately 0.95%. If the coating actually had a conductivity of 0.6 W/mC, the coating strain would increase to 1.50%, for the same heat flux. Thus, coating strain may be significantly affected by thermal conductivity, with the effect being more pronounced at higher thicknesses.

Figures 73 and 74 demonstrate the sensitivity of coating performance to thickness and material property variations. Thickness variations are significant due to the thin values required for high heat flux applications. Thus, although the variation required to cause failure may amount to a 100% increase in coating thickness, the absolute value may only be on the order of 0.01 mm. Control of coating thickness to this degree may be very challenging. Also, thermal conductivity measurements made earlier were found to vary by up to 15%, for one particular coating type. This variation may have a significant effect on coating performance. The point to be made is that the acceptable operating conditions for a particular coating system may be signifi-

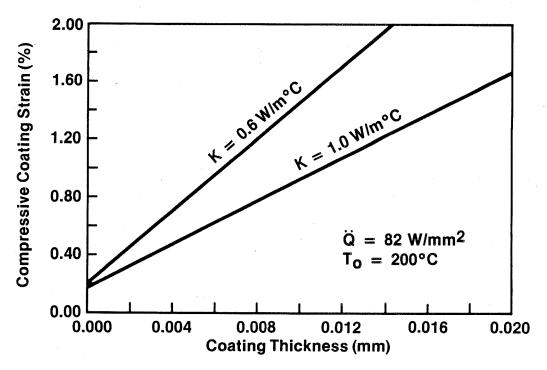


Figure 74. Effect of Coating Thickness on Maximum Compressive Coating Strain

cantly limited by the accuracy and precision of the coating application process.

The curves in Figures 73 and 74 were constructed for a specific heat flux. These curves are useful for preliminary coating system evaluations, if the heat flux is approximately correct. However, the true effect of TBC variations on coating performance requires an extended heat transfer analysis. For example, an increase in coating thickness will most likely cause a decrease in heat flux due to an increase in surface temperature. Thus, more detailed analyses are required to accurately evaluate a particular coating system for a specific application.

For example, a simple evaluation of the relationship between heat flux and coating thickness can be made as follows. Let us assume that the following parameters are constant: gas temperature (Tg), cold surface temperature (Tc), heat flow area (A), hot surface heat transfer coefficient (h), and the coating thermal conductivity (k). For a steady state thrust chamber condition, we can write:

$$Q/A = h(Tg - Ts) = k(Ts - Tc)/t$$
(33)

where t is the coating thickness.

Solving for t:

$$t = k(Ts - tc)/h(Tg - Ts)$$
 (34)

If we let ta represent the actual coating thickness and td represent the design coating thickness, we can evaluate the effect of thickness variations on the coating surface temperature:

$$ta/td = (Ts_a - Tc)(Tg - Ts_d/(Ts_d - Tc)(Tg - Ts_a)$$
 (35)

where the subscripts a and d represent actual and design conditions respectively. Assuming reasonable values for Tg (3000°C) and Tc (200°C) we can plot coating surface temperature versus normalized thickness (ta/td) for various design surface temperatures (see Fig. 75). Note that at a normalized thickness of 1.0, the actual coating surface temperature equals the design surface temperature. Actual surface temperature then increases non-linearly with coating thickness, rapidly at first, and gradually approaches a maximum value. It is interesting to note that coating surface temperature is especially sensitive to thickness variations around the design point.

Curves like those in Figure 75 can be used to select a nominal coating design thickness, based on a maximum allowable coating temperature and a probable thickness variation. By locating the point defined by (1) a normalized thickness based on the maximum possible coating thickness and (2) a maximum allowable coating temperature, one can define the coating design temperature and thus the nominal coating thickness. For example, if we select a maximum allowable coating temperature of 2000° C, and suspect the coating may be 100% thicker than the design thickness in some areas (Ta/Td = 2), then the coating should be designed for a surface temperature of approximately 1450%C.

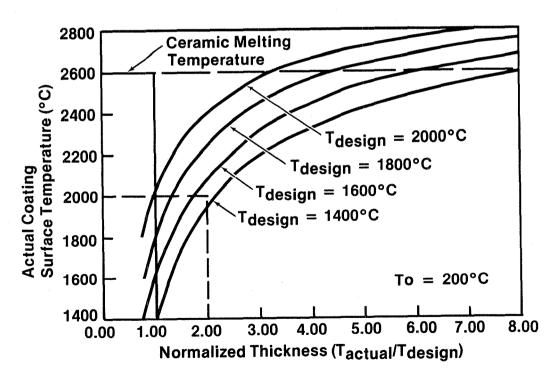


Figure 75. Effect of Thickness Variations on Surface Temperature

The analytical results shown in Figures 73 and 74 indicate that the TBC's under study must be very thin to survive the high heat fluxes anticipated in high pressure rocket engine thrust chambers. Figure 73 indicates that a coating thickness less than 0.015 mm is required to prevent surface melting at the highest heat flux evaluated. These analyses also indicate that the coatings under study are likely to fail structurally prior to melting. For example, Figure 73 shows that a 0.015 mm thick coating with $k=1.0\ \text{W/mC}$ would operate at the maximum desirable surface temperature of 1900°C . However, Figure 74 shows that this coating would incur a compressive strain of approximately 1.30 percent at this condition. This strain corresponds to a 10 to 50 percent survival probability, depending on which strength data set is used in Figure 66.

Figure 76 shows the effect of coating thickness on surface temperature for a coating with an average thermal conductivity of 0.935 W/mC, over a range of heat fluxes. This effect is linear for a constant heat flux. This analysis demonstrates that surface temperature is much more sensitive to coating thickness changes at high heat fluxes, than at low heat fluxes. The heat fluxes shown in this figure cover the range of anticipated operating conditions for a typical high pressure thrust chamber application. Again, it can be seen that a very thin coating (less than 0.02 mm) is required to accommodate

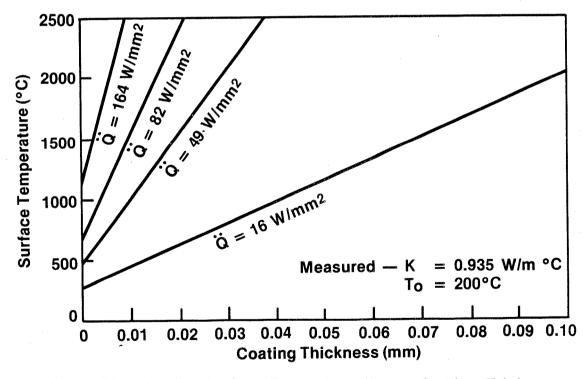


Figure 76. Coating Surface Temperature Versus Coating Thickness for Various Heat Fluxes Using Average Measured Thermal Conductivity

the high anticipated rocket engine heat fluxes (82 W/mm^2) without excessive surface temperatures (<2000°C) for this coating.

Figure 77 shows the effect of coating thickness on maximum compressive coating strain for the same conditions as just described. This effect is linear for a constant heat flux. Coating strain is most sensitive to coating thickness at the high heat flux conditions. At the highest anticipated rocket engine heat flux condition, 82 W/mm², the predicted coating strain is quite high compared to the survival probability curves shown in Figure 66. Coating thicknesses of this coating type which incur reasonable strains (say <1.0%)at this heat flux are quite thin (\sim 0.01 mm).

Figure 78 shows the effect of coating thickness and thermal conductivity on heat flux for a specified ceramic surface temperature of 1927°C and a cold surface temperature of -18°C . This hot surface temperature is a desirable operating limit for the ceramic coating. Heat flux is inversely proportional to coating thickness and directly proportional to thermal conductivity. The heat flux values shown in this figure cover the range of heat fluxes expected in most rocket thrust chambers. The coating thicknesses required to maintain an acceptable surface temperature at the most severe rocket engine conditions are again quite small (<0.02 mm), even for the coating with the highest measured thermal conductivity.

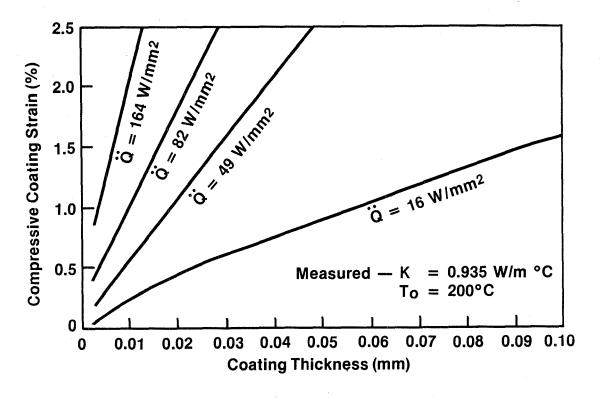


Figure 77. Coating Strain Versus Coating Thickness for Various Heat Fluxes

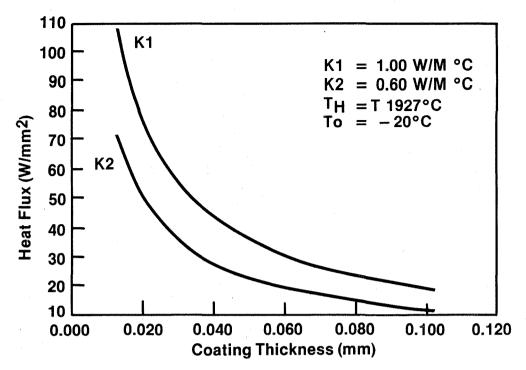


Figure 78. Heat Flux Versus Coating at Constant ΔT

Figure 79 shows the effect of coating thickness and temperature differential on the heat flux through a coating having an average measured thermal conductivity of 0.935 W/mc. The cold surface temperature for these analyses is -18°C. The ceramic surface temperature associated with the upper curve is 2200°C, which is greater than the desired maximum coating temperature, but less than the ceramic melting point. The lower curve represents a moderate ceramic surface temperature. Heat flux is inversely proportional to coating thickness and directly proportional to the temperature differential. The heat fluxes in this figure cover the range of values expected in the rocket engine thrust chambers. Again, the coating thicknesses required to accommodate the higher heat fluxes, even with an excessive ceramic temperature, under these conditions, is very small.

The performance curves presented in this section can be used to evaluate the general behavior of the TBC's produced in this study, under anticipated high pressure rocket thrust chamber operating conditions. These curves are also useful for demonstrating the effects of various parameters on coating performance. The analytical thrust chamber model can easily be used to evaluate a particular coating system under more specific operating conditions, and can be readily refined to incorporate more accurate material properties and/or boundary conditions as they become available.

3.4.3 Coating Design Technique

To illustrate an analytical coating design technique, the models described in this report were used to establish design criteria for the coatings produced

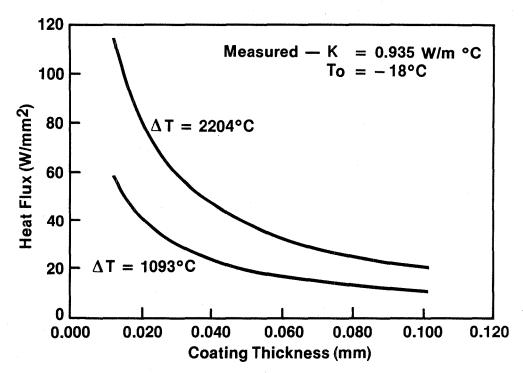


Figure 79. Heat Flux Versus Coating Thickness for Two ΔTs

in this study for rocket thrust chambers. These design criteria are essentially operating limitations and coating thickness limitations which must not be exceeded, to assure successful operation of the TBC. The physical limitations of the coating dictate the design criteria. The physical limitations considered for these analyses were maximum compressive coating strain and maximum coating temperature.

The maximum allowable coating strain was established using Figure 66. Strength data from the first, and weaker, specimen set were used to evaluate the "worst" case. A survival probability of 95 percent was arbitrarily selected as a design criterion for the coating. Using the curve in Figure 66 for data set 1, a maximum allowable coating strain of one percent was established. A coating temperature limitation of 2000°C was also selected, based on information from the literature and past experience at Solar.

The coating system model of a rocket thrust chamber wall was used to evaluate the performance of the TBC's under steady state operating conditions. Various operating conditions, coating thicknesses and coating types were evaluated to define those combinations that cause excessive strain or temperature in the coating. A constant cold surface temperature of 200°C was maintained for these analyses and a range of heat fluxes was investigated.

Two coating types, A and C, were evaluated. These represent the 20 and 40 kW coatings respectively. For these analyses, the coating types were differentiated only by the thermal conductivity of the coating. It was assumed that both coating types reflect the same strength distribution.

It was found that coating strain, rather than temperature, was always the limiting factor in these analyses. The results are presented in Figure 80 which shows maximum allowable heat flux through the coating vs. coating thickness for the two coating types. These results corroborate past experience which indicates that thin coatings can tolerate greater heat fluxes than thick coatings, without failing. These analyses show the inverse relationship between maximum allowable heat flux and coating thickness.

It is again noted that the high heat fluxes anticipated in high presure rocket engines (\sim 82 W/mm²) require an extremely thin coating (<0.02 mm) for structural integrity. Also of interest is the significant effect that coating properties and coating thickness can have on the allowable heat flux for thin coatings.

3.4.4 Discussion of Analytical Results

Coating systems analyses were conducted to establish the performance characteristics and operating limitations of the TBCs produced in this study for high heat flux applications. The finite element model that was used in these analyses simulated the coated throat section of a thrust chamber, which experiences an extremely high heat load. Previous calibration of the finite element model, including an evaluation of the significance of element aspect

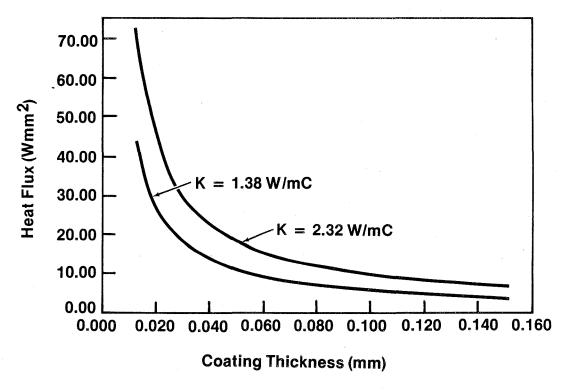


Figure 80. Maximum Heat Flux Versus Coating Thickness at 1% Strain

ratio, provides confidence that the model works properly. However, the accuracy of these results ultimately depends upon how closely the model simulates the actual coating system. This is determined largely by the accuracy of the geometry, boundary conditions and material properties.

The geometry of the finite element model is that of a cylinder. Although the actual thrust chamber throat is a curved surface, it is expected that the model geometry accurately represents a very small length at the middle of the throat, where the slope of the surface is parallel to the gas flow. The boundary conditions selected for these analyses covered ranges of anticipated operating conditions, based on the best information available from NASA. The most accurate material property data available were used in this model, and were specified as functions of temperature. The results of these analyses are therefore thought to be reasonably accurate.

These results suggest that the coatings evaluated in this study must be very thin to be practical for rocket thrust chamber applications. The thicknesses required to assure structural integrity at the high heat flux operating conditions are on the order of 0.010 mm. (This is based on the assumption that coating failure results from compressive thermal stress.)

Furthermore, it is obvious from these analyses that variations in coating thickness and material properties may significantly affect the performance of very thin coatings. Precise control over coating thickness and accurate knowledge of the coating properties may be essential to the successful operation of TBCs in high heat flux environments. Because of the thinness of these coatings, adequate control of the coating thickness may present a formidable challenge. Methods of accurately predicting coating material properties and precisely controlling coating thickness may each require a significant development effort.

Two basic approaches for making these coatings more compatible with the high pressure rocket thrust chamber requirements have been defined:

- 1. Develop a stronger coating to insure that surface temperature, rather than coating strain, is the primary limitation for heat flux. Then, by maintaining the lowest cold surface temperature practical, the heat flux through the coating can be maximized for a given coating thickness.
- 2. Develop a technique for controlling coating thermal conductivity. Then heat flux can be regulated within the material limitations of the coating system.

Both of these approaches require additional coating development work.

3.5 TEST THRUST CHAMBER COATING

A test thrust chamber is shown in Figure 81. This chamber has an inner diameter of 6.6 cm. Coating of this inner diameter required the use of different equipment than had been used in the previous program development

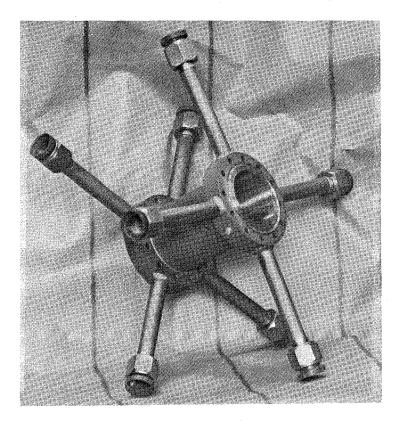


Figure 81. NASA Thrust Chamber for Test Coating

steps which employed a standard 7M gun configuration. The gun was changed to use a 7MBT-12 angle extension. The anode for this unit is designed to deflect the plasma gas at an angle of 45 degrees to the gun axis for coating the internal surfaces of small cylinders.

The equipment changeover required minor adjustments in the deposition process to achieve coatings equivalent to those initially established. Copper rings, 6.6 cm in diameter by 1.25 cm long, were fabricated from 0.81 mm copper sheet. The rings were mounted in a rotating fixture and the inner surfaces were coated. Forty-two rings were coated initially. Arbitrary adjustments to the gun parameters were made until coating appearance and deposit rates appeared similar to those obtained with the standard equipment. Initially, wide variations in coating quality were observed. These were traced to rapid anode wear. Normally these variations would not be significant when conventional coating thickness were being applied. However, the very thin coatings required for this application were sensitive to minor changes in gun characteristics. Anode usage was then limited to three hours.

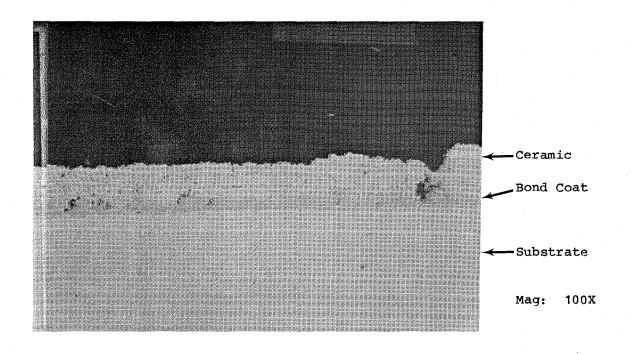
Twenty-eight additional rings were then coated and metallurgically sectioned to further refine the coating process. The parametric variations are given in Table 18. These tests included minor adjustments to the spray parameters and thickness variations to verify consistent coating application.

Table 18
Specimen Definition in Final Ring Coating Tests

		_	Thick (mil)		Gas flow (cfh)	
Ring Specime	en Volts-amps	Volts-amps	Bond	Ceramic	Bond	Ceramic
				_		
43	65-500	70-600	1	4	60	80
44	65-500	55-400	1	4 4 4 4 4 4	60	50
46	65-500	55-400	1	4	60	50
47	70-600	70-600	1	4	80	80
48	70-600	70-600	1	4	100	80
49	70-600	55-400	2 2 2	4	80	50
50	70-600	65-500	2	4	80	60
51	70-600	65-500			80	60
52	70-600	65-500	1	0.5	80	60
53	70-600	65-500	-1	0.5	80	60
54	70-600	65-500	1	0.5	80	60
55	70-600	65-500	1	1	80	60
56	70-600	65-500	1	1 1 2 2 2	80	60
57	70-600	65-500	1	1	80	60
58	70-600	65-500	1	2	80	60
59	70-600	65-500	1	2	80	60
60	70-600	65-500	1	2	80	60
61	70-600	N/A	1.5	N/A	70	N/A
62	70-600	70-600	1.5	1	70	80
63	70-600	70-600	1.5	0.5	70	80
64	70-600	70-600	1.5	2 4	70	80
65	70-600	70-600	1.5	4	-70	80
66	60-500	N/A	1.5	N/A	70	N/A
67	70-600	70-600	1.5	0.5	70	80
68	70-600	70-600	1.5		70	80
69	70-600	70-600	1.5	1 2	70	80
70	70-600	70-600	1.5	4	70	80
<u></u>						

The bond coat selected for these test chambers is shown in Figure 82. This bond coat was applied at 42 kW as indicated for ring #49 in Table 18. The ceramic applied to this ring was not considered to be suitable, so the ceramic coating parameters of ring 47, Figure 83, were selected for the 42 kW ceramic coating. Table 19 shows the parameters used to spray the cylindrical thrust chambers and the original flat test specimens.

The effect of increasing plasma gas flow and the bond coat can be seen by comparing Figure 82 and 83, both were applied at the same parameters except for plasma gas flow. The higher flow used for ring #47 caused overheating and oxidation of the substrate and poor deposition of the NiCrAly. However, for the zirconia coating it was found necessary to use a higher power level and gas flow to achieve the desired coating structure. The coating applied at 32 kW with a lower gas flow had a nonuniform structure with numerous large voids.



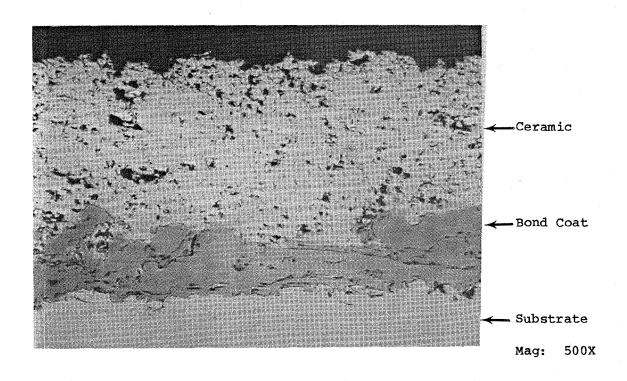
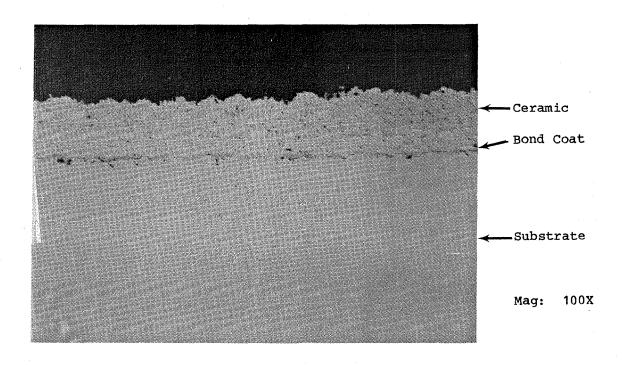


Figure 82. Photomicrographs of 42 kW Bond Coat (Ring 49)



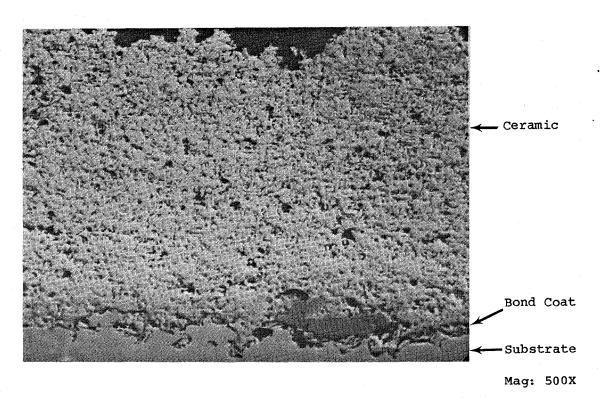


Figure 83. Photomicrograph of 42 kW Zirconia-8% Yttria Coating for Ring 47

Table 19
Plasma Spray Parameters for Flat and Ring Specimens

	Flat Ceramic Coating #1	, -	Flat Ceramic Coating #3	Flat Bond Coat	Ring Ceramic Coating	Ring Bond Coating
Current (amps) Voltage (volts) Power (kW) Argon (m³/hr) Hydrogen (m³/hr) Stand-Off (cm) Plasma Velocity (m/s) Power Port Cooling Air (kPa) Nozzle	400	500	600	500	600	600
	50	65	70	65	70	70
	20	32	42	32	42	42
	2.97	2.97	4.25	5.66	2.27	2.27
	0.085	0.14	0.20	0.11	0.425	0.453
	6.4	6.4	6.4	6.4	3.18	3.18
	25.4	30.5	43.7	32.5		
	No. 2	No. 2	No. 3	No. 2	No. 2	No. 2
	551	551	551	551	538	538
	GH	GH	703	GH	713	713

Having selected the specific parameters for coating the thrust chamber, a fixture was constructed for rotating and cooling the chamber during coating application. The set-up is shown in Figures 84 and 85. A standoff support was constructed to allow the plasma gun to exit from the chamber at each end of every traverse. This platform was then mounted on the end of a hollow shaft. Cooling air for the chamber was introduced into the hollow shaft through radial holes and a concentric manifold. This air was then introduced into the chamber fuel passages through tubing connected to the fuel lines, and used to cool the substrate during coating. A collector ring, not shown in the photographs, was added to catch the cooling air and direct it over a thermocouple for monitoring cooling air temperature during coating. The entire unit was then coupled to a variable speed drive for rotation during spraying. Rotational speed was 73.5 rpm which produced a surface speed of 15.2 m/min.

The first cylinder was coated to a nominal ceramic coating thickness of 0.025 mm. Telatemp Recorders (Telatemp, Fullertin, CA) were usd to determine the maximum substrate temperature during coating. A concentric flange with a 6.3 mm wall was attached to the end of the cylinder for metallurgical sectioning after coating. The cylinder was coated and a section made of the test specimen. It was found that the coating shown in Figure 86, was unacceptable. The directional effects of the 45 degree extension gun are evident. The coating section shows a shadowing effect from the initial particles much like a snow fence with inconsistent voids and uncoated areas. To prevent this, it was decided to reverse the cylinder with respect to gun traverse to provide a more uniform coating buildup. The effect of this on the ceramic phase is evident in Figure 87 with a uniform deposit of zirconia on each side of the bond coat projections. Backside cooling of the substrate was inadvertently reduced during this test and substrate overheating resulted in excessive oxidation and poor adherence.

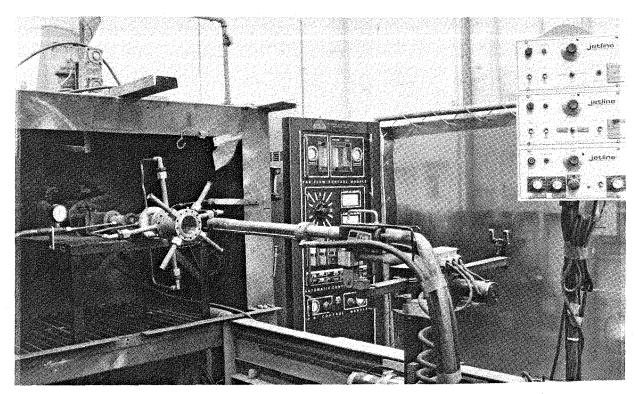


Figure 84. Overall Thrust Chamber Coating Set-Up

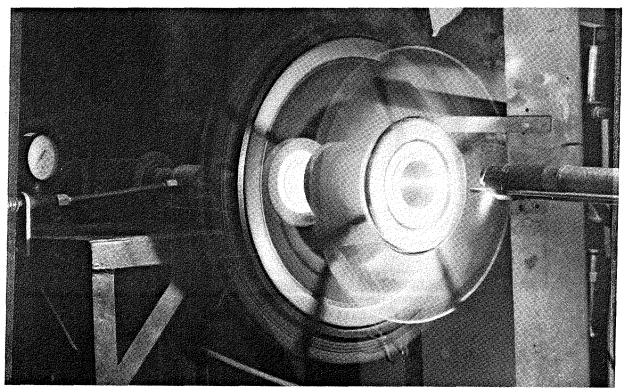
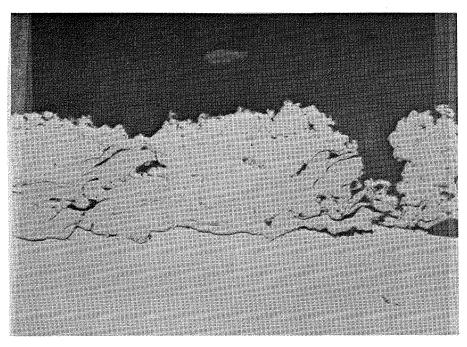
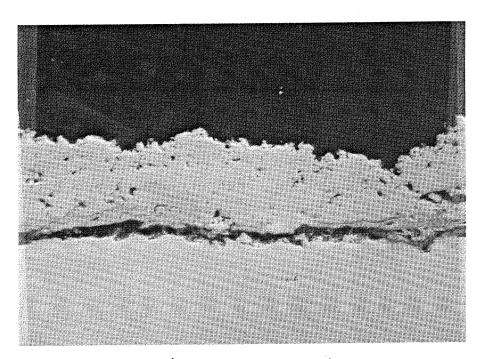


Figure 85. Coating Thrust Chamber



(Magnification: 500X)

Figure 86. First Coating Applied to a Thrust Chamber With a 7MBT Extension



(Mgnification: 500X)

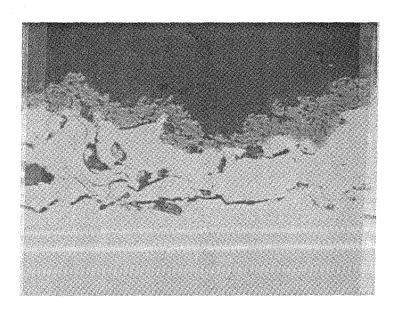
Figure 87. Coating Applied With Periodic Reversing of Cylinder

The initial cylinder was stripped and recoated with cyclic reversal. The maximum temperature of the unit was found to be 121°C using the data from the Telatemp recorders. A test section showed the coating thickness to be between 0.019 and 0.025 mm which is within the error expected for a nominal 0.025 mm coating.

The second cylinder was coated in the same manner to a nominal ceramic coating thickness of 0.013 mm. This coating thickness was found to range between 0.013 mm and 0.019 mm with the majority of the coating being close to the lower limit. This coating closely followed the contours of the bond coating without the leveling effects of the thicker coatings (see Fig. 88). It is expected that this might create a heat transfer problem in the anticipated service environment.

The third cylinder was coated to a thickness of 0.076 mm in the same manner as the previous two. No significant change in coating characteristics was observed. The fourth cylinder was coated to a thickness of 0.025 mm. This coating appeared to be slightly more porous than the previous specimens, but in the thin coating no quantitative differentiation could be made.

The change from flat specimens and rings, used for initial studies, to a small diameter cylinder introduced variations in the coating procedure. The limited access prevented using gun standoff as a process parameter and required adopting previously developed parameters in a new gun configuration. Also, parameters developed for gun power levels at significantly higher values than 40 kW could not be used.



(Magnification: 500X)

Figure 88. Coating Applied to a Planned Thickness of 0.013 mm

In practice, the coating power levels developed for the ceramic coating proved to be effective. It was necessary to adjust powder and plasma gas flow to provide a duplicate zirconia coating. Applying a reliable NiCrAlY bond coat was less successful and this process needs to be improved. The gases entrapped in the cylinder during spraying reacted to minor disturbances and unmelted oxidized particles were entrapped in the bond coat, preventing proper adherance. Bonding of the NiCrAlY to the copper substrate was not fully reliable and no nondestructive test was found. Throughout the cylinder coating tests this bond was found to be the weak point of the system and generated several problems in reliability that have not yet been resolved.

Four additional test chambers were coated at a later date. These coatings were applied with several process modifications to reduce substrate oxidation. The modifications consisted of improved substrate cooling, inert gas injection at the plasma gun tip, increased powder feed to reduce the time required to apply the bond coat, and improved substrate surface preparation using a larger grit size.

All of the coated test chambers are more fully documented in a separate lab report issued to the NASA program manager: Solar Report No. SR85-R-2151-00 (01/31/85).

4

CONCLUSIONS

- A practical analytical model of a TBC has been developed. This model combines empirical material property data with finite element analyses to predict the performance of the coating in service. This model can be used to evaluate the effect of coating deposition parameters on coating performance by first determining the effect of these parameters on the coating properties.
- 2. An analytical model of a porous coating structure was partially developed. This model requires additional development but appears to have potential for ultimately correlating coating thermal conductivity with deposition parameters.
- 3. Theoretical models of mixtures appear to have potential for predicting the thermal conductivities of TBC's. These models require empirical adjustment but may provide an accurate and practical method of calculating coating thermal conductivity as a function of porosity.
- 4. A practical method of testing coating strength has been developed. The four point flexure tests provide a simple and economical method of measuring coating strengths. Plastic deformation of the copper substrate causes complications in the strain analyses and contributes to the experimental error of the measurements. Due to the statistical nature of ceramic strengths, sets of 30 specimens are recommended per test to obtain reliable results.
- 5. A convenient method of utilizing the coating strength data is to establish a failure probability distribution for each coating type. Comparison of strength distributions can reveal differences between coating types. These failure probabilities can be used to statistically predict coating failure.
- 6. The coating stregth data generated in this study produced conflicting results. Two sets of data were generated at different times. The second set showed a significant increase in strength and a narrower strength distribution. Also, the first data set indicated an effect of plasma gun power level on coating strength whereas the second set did not. No conclusions are drawn as to the effect of power level or about the true strength of these coatings.
- 7. Performance analyses of the coatings produced in this study indicate that extremely thin (<0.02 mm) coatings may be required to operate reliably in the high heat flux environment of a high pressure rocket thrust chamber.

- 8. These performance analyses indicate that structural failure is likely to occur before the coating melts, for the coatings evaluated.
- 9. Two methods of improving the applicability of these coatings to rocket thrust chamber applications are (1) increase coating strength so that surface temperature becomes the limiting factor rather than strain and (2) increase the thermal conductivity of the coating to accommodate higher heat fluxes.
- 10. These analyses have demonstrated that coating performance is extremely sensitive to coating thickness and thermal conductivity at the very high heat flux conditions. The need for very accurate knowledge of the coating properties and precise control of the coating thickness is apparent at these high heat fluxes.
- 11. A practical coating design technique has been demonstrated. This technique uses the coating performance model to calculate the maximum allowable heat flux through the coating, as a function of thickness, based on physical coating limitations (strain, temperature). This provides a convenient plot of the recommended operating envelope for a specified coating reliability.
- 12. Coating tests indicate that the weak point in these coatings may be the bond-to-substrate interface.

5

RECOMMENDATION

- 1. Additional strength data should be generated to resolve the conflicting results obtained to date, and to establish a reliable failure probability distribution for these coatings. A method of eliminating plastic deformation of the substrate should be investigated to eliminate a source of experimental error. Testing specimens made with a stronger substrate (e.g., steel) should be tried.
- 2. A method of measuring Poisson's ratio and the elastic modulus of these coatings should be explored, to improve the basic coating model.
- 3. The porous coating model should be developed further. Alternative methods of characterizing a coating structure to accommodate analytical modeling should be explored.
- 4. Experimental coating development should be conducted to improve the strength of these coatings and perhaps to control their thermal conductivities. The analytical model of the porous coating may be useful in developing a technique for controlling the thermal conductivity of the coating.
- 5. Performance analyses of these coatings should be conducted for transient thermal conditions. These conditions may induce higher strains than those calculated for steady state conditions.
- 6. Coating strength should be measured as a function of time and temperature. This could be done by subjecting groups of specimens to high temperature environments for specified lengths of time and testing their strength with the four-point flexure test. These results could then be incorporated into the coating model to predict coating life during a specified operating cycle.

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APPENDIX 1

COATING STRESS/STRAIN ANALYSIS FOR FOUR POINT FLEXURE TEST

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Plastic deformation of the copper substrate during strength testing required that a sophisticated stress analysis be developed to calculate the true strains in the coating at failure. The premise of this analysis is that the neutral axis of the coated beam specimen shifts towards the ceramic coating as the copper plastically yields. The location of the neutral axis must be determined to evaluate the strain in the coating. The neutral axis is defined as the plane which experiences no axial dimensional change, thus the sum of the bending moments about this axis is zero. This provides a method of locating the neutral axis by using a moment balance.

It is assumed for this analysis that the copper substrate behaves as a perfectly plastic material (strain hardening is neglected) and possesses equal properties in tension and compression. Material property values for the copper and the ceramic were obtained from the literature (Refs. 10-16). These properties are:

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Yield strength, copper YS_{cu} = 70 MPa Elastic modulus, copper E_{cu} = 1.12 x 10^5 MPa Elastic modulus, ceramic E_{cer} = 4.48 x 10^4 MPa
```

The bond coat was neglected in these analyses.

The stress distribution in the coated flexure specimen during testing is shown in Figure 89. The stress in the copper substrate increases linearly with distance from the neutral axis, until the yield stress is reached. The stress beyond this point remains constant and equal to the yield strength of the copper. The strain in the ceramic coating is assumed to be elastic until the point of failure. Thus, the stress in the coating is assumed to remain directly proportional to the distance from the neutral axis.

Three dimensions are required for this analysis: R_1 (or R_2), t_1 and t_2 which are defined in Figure 89. An arbitrary approximation for the radius of curvature of the neutral axis (R) is used to begin an iterative solution procedure. Each iteration involves calculating the extent of the elastic zone (e) and the net moment about the neutral axis. If the calculated net moment is not sufficiently close to zero, a new value of R is selected and the next iteration performed. The true neutral axis is found by iterating until the net moment is essentially zero.

The computer program created for this analysis uses the midpoint of the beam specimen, $(t_1+t_2)/2$, as the initial "guess" for R. An iteration procedure is then used to converge on the true R, subtracting increments of DR (typically 2.54 x 10^{-6} mm) from R for each iteration. Convergence on a solution is determined when the sense of the net moment changes sign. The true neutral axis is calculated as the average of the last two radii evaluated. The strain in the ceramic coating ($\epsilon_{\rm C}$) is then calculated from

$$\varepsilon_{\rm c} = {\rm c/R}$$

where c is the distance from the neutral axis to the point of interest.

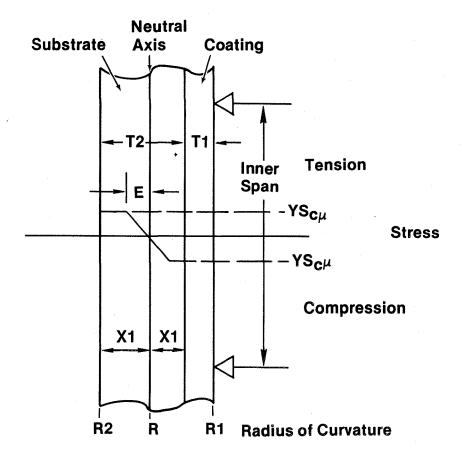


Figure 89. Stress/Strain Model for Plastically Deforming Four-Point Bend Specimen

APPENDIX 2

EXAMPLES OF FINITE ELEMENT THRUST CHAMBER ANALYSES INPUT AND OUTPUT
(THERMAL AND STRESS)

INTRODUCTION

This appendix documents the basic thermo-mechanical coating system computer analyses used to evaluate thrust chamber coating performance on this program.

Representative input and output files for use with the ANSYS computer code are presented. A thermal analysis input file is listed on page 121. The associated output for this file is listed on pages 122 through 136.

A stress analysis input file is listed on page 138. The associated output for this file is listed on pages 139 through 154.

Post processing input and output files are listed on pages 155 and 156. This analysis calculates stresses at the nodes in the finite element mode.

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         FILL
         N.57.1.3085556..009
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          FILL
         N.64.1.314.009
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         N.71.1.3202222.009
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         FILL
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         FILL
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         MAT.1
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         EGEN.3.7.1.6
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         EGEN. 6.1.19
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         EGEN.3.7.19.24
MAT.3
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         E.44.51.50.43
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         EGEN.6.1.37
EGEN.6.7.37.42
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         ITER. -10.10
         TIER.-10.10
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KHC.1
HFLOW.1.HEAT.00.0097088..7.6
HFLOW.2.HEAT.0.0194175..6.1
NT.85.TEMP.400...91
AFWRIT.1.1
FINISH
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          /INRUT.27
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 62
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SWANSON ANALYSIS SYSTEMS. INC.
                                                STEM REVISION 4.0 E2 SCLAR TURBINES AUG 1,1981
HOUSTON. PENNSYLVANIA 15342 PHONE (412)746-3304 TWX 51
                                    ANALYSTS SYSTEM
  TITLE
                                                                                          10.0986
           **** ANSYS ANALYSIS DEFINITION (PREP7) ****
 NEW TITLE THRUST CHAMBER MODEL
PRINTOUT RESUMED BY /GOP
 ANALYSIS TYPE= -1
 TEMPERATURE OFFSET FROM ABSOLUTE ZERO= 460,000
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KEYOPT(1-9) = 2 0 1
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TWO-DIMENSIONAL STRUCTURE .
  MATERIAL
                          COEFFICIENTS OF KXX VS. TEMP EQUATION
   c_0 = 0.1250000E-04
                       MAT= 1 NUM POINTS= 2
DATA TEMPERATURE
 PROPERTY TABLE KXX
      TEMPERATURE DATA TEMPERATURE DATA 0.0000000E+00 0.1250000F+04 2300.000 0.1250000E+04
   ATERIAL 2 COEFFICIENTS OF KXX VS. TEMP EQUATION CO = 0.1180000E-03 C1 = 0.1180000E-07 C2 = 0.0000000E+00 C3 = 0.0000000E+00 C4 = 0.0000000E+00
  MATERIAL
 MATERIAL 3
CO = 0.5231400E-02
                          CUEFFICIENTS OF KXX VS. TEMP EQUATION
PROPERTY TABLE, KAX WAT= 3 NUM. POINTS= 2
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                     MAT= 2 NUM_ POINTS= 2
DATA TEMPERATURE
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5/ 8/84 C#

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NODE 7 KCS= 0 X,Y,Z= 1.2945 0.00000E+00 0.00000E+00
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        50 KCS= 0 X,Y,Z= 1.3639
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NODE 56 KCS= 0 x.Y.Z= 1.3039 .0.00000E+00.0.00000E+00
   TLL 5 POINTS BETWEEN NODE 50 AND NODE START WITH NODE 51 AND INCREMENT BY 1
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NODE 57 KCS= 0 X.Y.Z= 1.3086
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          63 KCS= 0 X.Y.7= 1.3086
 NODE
                                               C_00000F+00 0_00000E+0u
  START WITH NODE 58 AND INCREMENT BY 1
 FILL
 NODE 64 KCS= 0 x,Y,Z= 1.3140 0.90000E-02 0.00000E+00
 NODE
         70 KCS= 0 X.Y.Z= 1.3140
                                               0,00000F+00 0,00000E+00
  START WITH NODE 65 AND INCREMENT BY 1
          71 KCS= 0 X.Y.Z= 1.3202
 NODE
                                               0.90000E=02 0.00000E+00
 NODE 77 KC3= 0 X.Y.Z= 1.3202
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FILL 5 POINTS BETWEEN NODE 71 AND NODE START WITH NODE 72 AND INCREMENT BY 1
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          78 KCS= 0 X.Y.Z= 1.3272
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NODE
         84 KCS= 0 X.Y.Z= 1.3272
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  TILL 5 POINTS BETWEEN NODE 78 AND NODE START WITH NODE 79 AND INCREMENT BY 1
NODE 85 KCS= 0 X,Y,Z= 1.3350
                                             NODE
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 GENERATE 6 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF SET IS 19 TO 19 IN STEPS OF 1 NUMBER OF ELEMENTS = 24
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THEGER STORAGE REQUIREMENTS FOR ELEMENT INPUT

MEMORY I= 1001 MEMORY II= 182 TUTAL= 1183 MEMORY AVAILABLE= 340000

MAXIMUM NODE NUMBER FOR AVAILABLE AUXILIARY MEMORY SIZE= 169498

NUMBER OF ELEMENTS =

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72

MAXIMUM NODE NUMBER USED =

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<u>5</u>			00000F+00 00000F+00		
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8 9			00000F+00 00000F+00		
10	1.2958 0.60	000E-05 0.	00000F+00		
11 12	1.2958 0.45 1.2958 0.30		00000F+00 00000F+00		
13	1.2958 0.15	000E-02 0.	0000F+00		
14 15			00000F+00 00000F+00		
16	1.2972 0.75		00000F+00		
17	1-2972 0-60		00006F±06		
18			00000F+U0 00000F+U0		
20	1,2972 0,15	000E-02 0.	00000F+00		
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34			00000F+60 00006F+00		
34	1.2990 0.15	000F-02 ~0.	00000F+00		
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37	1.2995 0.75	000E=02 0.	000G0F+G0		
39 39			00000F+00		
40	1.2995 0.30		00000F+00 00000F+00		
41	1.2995 0.15	000E-02 0.	00000F+00		
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64	1.3140 0.90	000E-02 0-	00000E+00		
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87 88		00E-02 0.	00000E+00		
89	1.3350 0.300		00000F+00 00000F+00		
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YMIN= 1.295 XMAX= 1.335 YMIN= 0.0000E+00 YMAX= 0.9000E-02 ZMIN= 0.0000E+00

TNTEGER STORAGE REGUIREMENTS FOR NODE INPUT

MEMORY I= 0 MEMORY II= 546 TUTAL = 546 MEMORY AVAILABLE= 340000

MAXIMUM NODE NUMBER FOR AVAILABLE AUXILIARY MEMORY SIZE= 113333

```
**** MATERIAL PROPERTIES ****
 MATERIAL
 KXX PROPERTY TABLE (LINEAR INTERPOLATION)
TEMP KXX
 0.0 0.12500E-04 2300.0 0.12500E-04
 MATERIAL
 KXX PROPERTY TABLE (LINEAR INTERPOLATION) TEMP KXX
0.0 0.11800E-03 2300.0 0.26014F-03
 MATERIAL
 KXX PROPERTY TABLE (LINEAR INTERPOLATION)
TEMP KXX
0.0 0.52314E-02 2300.0 0.52314F-02
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 C PROPERTY TABLE (LINEAR INTERPOLATION)
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 MATERIAL 1
THE RESERVE THE SECOND STREET SECOND STREET
  DENS PROPERTY TABLE (LINEAR INTERPOLATION)
EMP DENS TEMP DENS
0.0 1.0000 2300.0 1.0000
 MATERIAL
 DENS PROPERTY TABLE (LINEAR INTERPOLATION)
TEMP DENS TEMP DENS
0.0 1.0000 2300.0 1.0000
 MATERIAL
 DENS PROPERTY TABLE (LINEAR INTERPOLATION)
TEMP DENS TEMP DENS
0.0 1.0000 2300.0 1.0000
MAXIMUM MATERIAL NUMBER= 3
```

**** MASTER DEGREES OF FREEDUM ****

NODE

DEGREES OF FREEDOM LIST

NUMBER OF SPECIFIED MASTER D.O.F. = 0 TOTAL NUMBER OF MASTER D.O.F. = 0
TNTEGER STORAGE REQUIREMENTS FOR MATERIALS, ETC. THRUT CP= 16.940 TIME= 10.1082 MEMORY I= 92 MEMORY II= 0 TOTAL= 92 MEMORY AVAILABLE= 340000
*** LOAD STEP 1 OFTIONS SPECIFICATIONS
NTTTER= -10 NPRINT= 10 NPOST= 10
ALL PRINT CONTROLS RESET TO 10
ALL POST DATA FILE CONTROLS RESET TO 10
ACEL= 0.00000E+00 0.00000E+00 0.00000E+00
OMEGA= 0.00000E+00 0.00000E+00 0.00000F+00
DOMEGA= 0.00000E+00 0.00000E+00 0.00000F+00
CGLOC= 0.00000E+00 0.00000E+00 0.00000E+00
CGOMEGA= 0.00000E+00 0.00000E+00 0.00000F+00
DOMEGA= 0.00000E+00 0.00000E+00 0.00000F+00
KTEMP= 0 0
ALL TEMPERATURES SET TO TUNIF = 0.000
STEADY STATE CONVERGENCE CRITERION=, 1,0000 TRANSIENT UPTIMIZATION CRITERIA= 5,0000 TEMPERATURE LIMIT= 0,0000E+00
KFY TO TERMINATE RUN IF NO CONVERGENCE = 0.
TOADS STEPPED TO FINAL VALUES FOR ALL ITERATIONS (KBC= 1)
REACTION FORCE KEY= 1
UNIFURM TEMPERATURE = 0.000 (TREF= 0.000)
ROUNDARY CONDITION PRINT KEY= 0

```
I OAD STEP NUMBER = 1
      *** LOAD OPTIONS SUMMARY
 TIME = 0.00000E+00 NITTER=
                     = 0.00000E+00 0.00000E+00 0.00000E+00
ACEL = 0.00000E+00 0.00000E+00 0.00000F+00 0.00000F+00 0.00000E+00 0.00000E+00 0.00000F+00 0.00000F+00 0.00000E+00 0.0000E+00 0.0000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0
                                                                                                                              O ITERATION=
                                                                                       0.000 (TREF=
  UNIFORM TEMPERATURE=
                                                                                                                                                    0.000)
 PRINT ELEMENT FORCES AND REACTION FORCES (KRF=1)
 I DADS STEPPED TO FINAL VALUES FOR ALL ITFRATIONS (KBC= 1)
STEADY STATE CONVERG CRII = 1.0000
TRANSIENT UPT CRITERIA= 5.0000
TFMPERATURE LIMIT= 0.00000±+00
KFY TO TERMINATE RUN IF NO CONVERGENCE= 0.
 MODE O ISYME
     IPRINT= 10 NPOST= 10 REACTION PRINT FREQ=
DISP_ POST DATA FREQ= 10 REACT_ POST DATA FREQ=
     DISPLACEMENT PRINT FREULENCIES
FREU NSTRT NSTOP NINC
10 1 32000 1
     FLEMENT PRINT AND POST DATA FREQUENCIES
TYPE STIFF SIRESS FORCE STRESS DATA
NC. PRINT PRINT DATA LEVEL
                                                                                                                                                DATA FORCE
LEVEL DATA
3 10
                                                      10 10 10
     **** SPECIFIED TEMPERATURES *****
             NODE
                                                TEMP
                                      400-000
400-000
400-000
             85
             86
87
                                      400.000
             8.8
              89
                      400.000
             90
             91
                                      400.000
                                          **** SPECIFIED HEAT FLOW RATES ****
                                                                                                                              VALUF
             NO.
                                      NODE
                                                                      D. OF FR.
                                                                             HEAT
                                                                                                          0.970880E-02
                                                                             HEAT
                                                                                                          0.970880E-02
0.194175E-01
0.194175E-01
                                                                             HEAT
                                                                                                          0.194175E-01
0.194175E-01
                                                                             HEAT
                                                                                                          0.194175E-01
**** LOAD SUMMARY - 7 TEMPERATURES 7 HEAT FLOWS 0 CONVECTIONS
  INTEGER STORAGE REGLIREMENTS FOR LOAD DATA INPUT
MEMORY I= 216 MEMORY II= 0 TOTAL = 216 MEMORY AVAILABLE= 340000
```

RANGE OF ELEMENT MAXIMUM CONDUCTIVITY IN GLOBAL COORDINATES MAXIMUM= 0.125014E-01 AT ELEMENT 72 MINIMUM= 0.103696E-04 TAT ELEMENT 6

```
THIEGER STORAGE REQUIREMENTS FOR ELEMENT FORMON ACTION

MEMORY I= 214 MEMORY II= 0 TOTAL= 214 MEMORY AVAILABLE= 340000
   *** FLEMENT STIFFNESS FORMULATION TIMES TYPE NUMBER STIF TOTAL CP AVE CP
    1 72 55 4.550 0.063
 TIME AT END OF ELEMENT STIFFNESS FORMULATION CP =
     MAXIMUM IN-CORE WAVE FRONT ALLOWED FOR REQUESTED MEMORY SIZE= 580
                                                                                                CP=
 TRIEGER STURAGE REQLIREMENTS FOR WAVE FRONT MATRIX SOLUTION CP= 29.730 TIME= 10.11592
MEMORY I= 214 MEMORY II= 121 TOTAL= 335 MEMORY AVAILABLE= 340000
 MAXIMUM IN-CORE WAVE FRONT (EQUATIONS) USED= 9
  *** MATRIX SOLUTION TIMES
READ IN ELEMENT STIFFNESSES
NODAL COORD. TRANSFORMATION
MATRIX TRIANGULARIZATION
                                                CP=
CP=
  TIME AT END OF MATRIX TRIANGULARIZATION
  TNTEGER STORAGE REQLIREMENTS FOR BACK SUBSTITUTION CP= 30.180 TIME= 10.1161 MEMORY I= 214 MEMORY II= 186 TOTAL= 400 MEMORY AVAILABLE= 340000
*** ELEM. HT. FLUN CALC. TIMES
TYPE NUMBER STIF TOTAL CP AVE CP
                                  2.390 0.033
               72
                        55
   *** NODAL HT. FLOW CALC. TIMES
TYPE NUMBER STIF TOTAL CP AVE CP
     1 72 55 0.260 0.004
   *** LOAD STEP 1 ITER 1 COMPLETED. TIME= 0.000000E+00 KUSE= 0 CUM. ITER.=
  TNTEGER STORAGE REQLIREMENTS FOR HEAT FLOW CALCULATIONS CP= 33.890 TIME= 10.11827
WEMORY I= 214 MEMORY II= 364 TOTAL= 578 MEMORY AVAILABLE= 340000
 *** STURAGE REQUIREMENT SUMMARY
MAXIMUM CENTRAL MEMORY USED = 1001
MAXIMUM AUXILIARY MEMORY USED = 546
MAXIMUM TOTAL MEMORY USED = 1183
MAXIMUM AUXILIARY AVAILABLE = 339784
                                                 339784
   *** PROBLEM STATISTICS
NO. OF ACTIVE DEGREES OF FREEDOM = 84
R_M_S_ WAVEFRONT SIZE = 7.5
 *** ANSYS RINARY FILE STATISTICS
RUFFFR SIZE USED= 2040
POST DATA WRITTEN ON FILE 12
RESTART DATA WRITTEN ON FILE 3 ( 184
TEMPERATURES WRITTEN ON FILE 4
                                               ( 18426 WORDS)
*** LOAD STEP 1 ITER 2 COMPLETED' TIME= 0'000000E+00 KUSE= 0 CUM. ITER.= 2
STEADY STATE CUNVERGENCE VALUE = 27.4601 AT NODE 22 CRITERION = 1.0000
   *** LOAD STEP 1 ITER 3 COMPLETED. TIME = 0.000000E+00 KUSE D CUM. ITER. = STEADY STATE CONVERGENCE VALUE = 0.5384 AT NODE ?? CRITERION = 1.0000
          OLUTION CONVERGED - LOAD STEP 1 CONVERGED AFTER ITERATION 3 CUM. ITER.= 3
NEXT ITERATION (IDENTIFIED AS ITERATION 10) SATISFIES PRINTOUT OR POST DATA REQUEST.
   *** SOLUTION CONVERGED - LOAD STEP
```

**** TEMPERATURE NODE TEMP 1 3760.5 6 3760.5 11 2694.7 16 1629.1 21 1629.1	SOLUTION **** NODE 2 7 12 17 22	* TIMF : TEMP 3760 5 3760 5 2694 7 1629 1 565 47	= 0.00000 NODE 3 4 13	F+00 LOAD S TEMP 3760.5 2694.7 2694.7 1629.1 565.47	TEP= 1 , NODE 4 9 14 19	ITERATION= TEMP 3760.5 2694.7 2694.7 1629.1 565.47	10 CUM. NODE 5 10 15 20 25	175 376 369 162 162 565
26 565.47 31 532.67 36 499.43 41 499.43 46 465.74 51 458.35 61 449.51	27 32 37 42 47 52	565-47 532-67 499-43 465-74 458-35 449-51	, 28 338 343 443 54 54 63	565.47 532.67 499.43 465.74 465.74 458.35 449.51	29 34 39 44 49 54 59 64	532-67 532-67 499-43 465-74 465-74 458-35 449-51 439-23	30 35 40 50 50 60	157395888 157465888 4459 4439
66 439.23 71 427.54 76 427.54 81 414.46 86 400.00 91 400.00	67 72 77 82 87	439.23 427.54 427.54 414.46 400.00	68 73 78 83 88	439.23 427.54 414.46 414.46 400.00	69 74 79 84 89	439.23 427.54 414.46 414.46 400.00	70 75 80 85 90	439 427 414 400 400
MAXIMUM TEMPERATU MINIMUM TEMPERATU		AT NODE	1 91		WINDOW ST . PROSECUTION OF THE PROSECUTION			· constant
**** ELEMENT HEAT	FLOW RATES **	*** TIME =	0.000000F	+00 LUAD S	TEP= 1	ITER.=	10 CUM.	ITER.
HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	9 0.97087 8 0.97987	8E-02 6F-02						
FLEMENT 2 HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	10 0.97087 9 0.97087	6E-02 5E-02	and the second of the second of				ante (18 caption	
FLEMENT HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	11 0.97087 10 0.97087	5E-02 5E-02					engen en e	
FIEMENT 4 HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	12 0.97087 11 0.97087	5F-02					e and a second s	
FLEMENT S HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	13 0-97087 12 0-97087	5E-02 6E-02					Andrew Marie State	
ELEMENT HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE	14 0.97087 13 0.97087	6E-02					allen or stronger comments	
FLEMENT 7 HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE	16 0.97087 15 0.97087	6E-02			 			
FLEMENT 8 #FAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE	17 0.97087 16 0.97087	6F-02 6E-02				to the second second		
FLEMENT 9 HEAT FLOW INTO NODE	11-0.97087	5F-02						

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HEAT FLOW INTO MODE
HEAT FLOW INTO MODE
HEAT FLOW INTO MODE
                                                                                   16 0.9708755-02
17 0.9708756-02
10 -0.970876E-02
    FLEMENT
   HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                           12 -0.9708/6F-62
19 6.970675F-02
18 0.970876E-02
11 -0.970875E-02
   FIEMENT 11
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                    13 -0.970876F-02
20 0.970676F-02
19 0.970876E-02
12 -0.970875E-02
   FIEMENT 12
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                   14 -0.970876E-02
21 0.970876E-02
20 0.970876E-02
13 -0.970876F-02
FIEMENT 13
HFAT FLOW INTO HODE
HFAT FLOW INTO HODE
HFAT FLOW INTO HODE
HFAT FLOW INTO HODE
                                                                                   16 -0.970876E-02
23 0.970876E-02
22 0.970876E-02
15 -0.970876E-02
    FLEMENT
   HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             17 -0.970876E-02
24 0.970876F-02
23 6.970876E-02
16 -0.970876E-02
FLEMFNT 15
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                            18 -0.970876F-02
25 0.970876F-02
24 0.970676F-02
17 -0.970876F-02
   FIEMENT 16 NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE
                                                                          19 -0.970876F-02
26 0.970876F-02
25 0.970876F-02
18 -0.970876F-02
   FIEMFNT 17
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                  FIEMFNT 18
HFAT FLOW 1NTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                   21 -0.970876F-02
28 0.970876F-02
27 0.970876E-02
20 -0.970876F-02
   FIEMENT 19
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                   23 -0.970876E-02
30 0.970876E-02
29 0.970876E-02
22 -0.970876E-02
   FIEMENT 20
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                    24 -0.970876E-02
31 0.970876E-02
30 0.970876E-02
23 -0.970876E-02
    FLEMENT
  HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                   25 -0.970876E-02
32 0.970876E-02
31 0.970876E-02
24 -0.970876E-02
   FLEMFNT 22
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                    26 -0.970876E-02
33 0.970876E-02
32 0.970876E-02
25 -0.970876E-02
    FI EMENT
  FLEMENT 23
HEAT FLOW INTO NODE
                                                                                    27 -0.970876E-02
```

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HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE
                                                                                        34 0.970876F=02
33 0.970876E-02
26 -0.970876E-02
    FIEMENT 24
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                        28 -0.970876E-02
35 0.970876E-02
34 0.970876E-02
27 -0.970876E-02
    FLEMENT 25
HFAT FLUW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                        30 -0.970876E-02
37 0.970876E-02
36 0.970876E-02
29 -0.970876E-02
    FLEMENT 26
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                        31 -0.970876E-02
38 0.970876E-02
37 0.970876E-02
30 -0.970876E-02
   FIEMENT 27
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                        32 -0.970876E-02
39 0.970876E-02
38 0.970876E-02
31 -0.970876E-02
   FLEMENT 28
HFAT FLOW 1NTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                       33 -0.970876E-02
40 0.970876F-02
39 0.970876E-02
32 -0.970876E-02
 FIEMENT 29
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NOCE
                                                                                   34 -0.970876E-02
41 0.970876E-02
40 0.970876F-02
33 -0.970876F-02
   FIEMENT 30
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                               35 -0.970876E-02
42 0.970876E-02
41 0.970876F-02
34 -0.970876E-02
   HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                    37 -0.970876E-02
44 0.970876E-02
43 0.970876E-02
36 -0.970876E-02
FLEMENT 32
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                       38 -0.970876E-02
45 0.970876E-02
44 0.970876E-02
37 -0.970876E-02
   FILMENT 33
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                       39 -0.970876E-02
46 0.970876E-02
45 0.970876E-02
38 -0.970876E-02
     FI EMENT
   HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                       40 -0.970876E-02
47 0.970876E-02
46 0.970876F-02
39 -0.970876E-02
   HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                    41 -0.970876E-02
48 0.970876E-02
47 0.970876E-02
40 -0.970876E-02
    FI EMENT
   HFAT FLOW INTO NODE 42 -0.970876E-02
HFAT FLOW INTO NODE 49 0.970876E-02
HFAT FLOW INTO NODE 48 0.970876E-02
HFAT FLOW INTO NODE 41 -0.970876F-02
FLEMENT 37
THEAT FLOW INTO NODE 44 -0.970876F-02
```

```
FLEMENT 38
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                   45 -0.970676E-02
52 0.970876E-02
51 0.970876E-02
44 -0.970876E-02
  FLEMFNT 39
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                            46 -0.970676E-02
53 6.970876F-02
52 0.970876F-02
45 -0.970676E-02
  FLEMENT 40
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                         47 -0.970876E-02
54 0.970876F-02
53 0.970876F-02
46 -0.970876F-02
 FLEMENT 41
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                            48 -0.970876E-02
55 0.970876E-02
54 0.970876E-02
47 -0.970876E-02
   FIEMENT
  HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE
                                                                             49 -0.970876E-02
                                                                    56 0.970876E-02
55 0.970876E-02
48 -0.970876E-02
   HEAT FLOW INTO NODE
 FIEMENT 43
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                           51 -0.970876F-02
58 0.970876F-02
57 0.970876F-02
50 -0.970876E-02
  FIEMENT 44
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             52 -0.970876F-02
59 0.970676F-02
58 0.970876F-02
51 -0.970876F-02
  FLEMENT 45
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                          53 -0.970876E-02
60 0.970876F-02
59 0.970876E-02
52 -0.970876E-02
FLEMENT 46
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                            54 -0.970876F-02
61 0.970876F-02
60 0.970876E-02
                                                                             53 -0.970876E-02
  FLEMENT 47
HFAT FLOW INTO NGDE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             55 -0.970876E-02
62 0.970876E-02
61 0.970876E-02
54 -0.970876E-02
 FLEMENT 48
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             56 -0.970876E-02
63 0.970876E-02
62 0.970876E-02
55 -0.970876E-02
  ELEMENT
  HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             58 -0.970876E-02
65 0.970876E-02
64 0.970876E-02
57 -0.970876E-02
 FIEMENT 50
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                             59 -0.970876E-02
66 0.970876E-02
65 0.970876E-02
58 -0.970876E-02
FLEMENT - 51
HEAT FLOW INTO NODE
                                                                             60 -0.970876F-02
```

```
HEAT FLOW INTO NODE HEAT FLOW INTO NODE HEAT FLOW INTO NODE
                                                                                       67 0.970876E-02
66 0.970876F-02
59 -0.970876E-02
 FLEMFNT 52
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                 61 -0.970876E-02
68 0.970876E-02
67 0.970876E-02
60 -0.970876E-02
FLEMENT 53
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      62 -0.970876E-02
69 0.970876E-02
68 0.970876E-02
61 -0.970876E-02
   FLEMENT 54
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                      63 -0.970876E-02
70 0.970876E-02
69 0.970876E-02
62 -0.970876E-02
FIEMENT 55
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      65 -0.970876E-02
72 0.970876E-02
71 0.970876E-02
64 -0.970876E-02
   FLEMENT 56
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      66 -0.970876E-02
73 0.970876E-02
72 0.970876E-02
65 -0.970876E-02
 FLEMENT
   HEAT FLUW INTO NUDE
HEAT FLOW INTO NUDE
HEAT FLOW INTO NUDE
HEAT FLUW INTO NUDE
                                                                                      67 -0.970876E-02
74 0.970876E-02
13 0.970876E-02
66 -0.970876E-02
    ELEMENT
                                          58
   HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE HFAT FLOW INTO NODE
                                                                                      68 -0.970876E-02
75 0.970876E-02
74 0.970876E-02
67 -0.970876E-02
FLEMENT 59
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      69 -0.970876E-02
76 0.970876E-02
75 0.970876E-02
68 -0.970876E-02
  FLEMENT 60
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      70 -0.970876E-02
77 0.970876E-02
76 0.970876E-02
69 -0.970876E-02
   FLEMENT 61
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      72 -0.970876E-02
79 0.970876E-02
78 0.970876E-02
71 -0.970876E-02
  FIEMENT 62
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                       73 -0.970876E-02
80 0.970876E-02
79 0.970876E-02
72 -0.970876E-02
   FLEMENT 53
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                      74 -0.970876F-02
81 0.970876E-02
80 0.970876E-02
73 -0.970876E-02
    FLEMENT
   FLEMENT 64
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                75 -0.970876E-02
82 0.970876E-02
81 0.970876E-02
74 -0.970876E-02
   FLEMENT 65
HEAT FLOW INTO NODE 776 -01970876E-02
```

```
HEAT FLOW THIC NODE HEAT FLOW INTO WODE
                                                                                            05 0.9706765-02
62 0.9706765-02
75 -0.9706765-02
 FLEMFNT BB
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                            77 -0.970876E-02
84 0.970876E-02
83 0.970876E-02
76 -0.970876E-02
 FIEMENT 67
HEAT FLOW INTO NUDE
HEAT FLOW INTO NUDE
HEAT FLOW INTO NUDE
HEAT FLOW INTO NUDE
                                                                                            79 -0.970876E-02
86 0.970876E-02
85 0.970876E-02
78 -0.970876E-02
      FIEMENT 68
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                           80 -0.970876E-02
87 0.970876E-02
86 0.970876E-02
79 -0.970876E-02
  FIEMENT 69
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                           81 -0.970876E-02
88 0.970876E-02
87 0.970876F-02
80 -0.970876F-02
       FLEMENT
                                             70
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
HEAT FLOW INTO NODE
                                                                                          82 -0.970876E-02
89 0.970876E-02
88 0.970876E-02
81 -0.970876E-02
FLEMENT 71
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                          83 -0.970876E-02
90 0.970876E-02
89 0.970876E-02
82 -0.970876E-02
    FLEMENT 72
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
HFAT FLOW INTO NODE
                                                                                          84 -0.970876E-02
91 0.970876F-02
90 0.970876F-02
83 -0.970876F-02
```

ANSYS - ENGINEERING ANALYSTS SYSTEM REVISION 4.0 E2 SOLAR SWANSON ANALYSIS SYSTEMS, INC. HOUSTON, PENNSYLVANIA 15342 PHO	FURBINES AUG ONE (412)746-3	1,1981 304 TWX 510
THRUST CHAMBER MODEL	10.1357	5/ 8/84 CF
***** NODAL HEAT FLOW RATES ***** TIME = 0.00000E+00 LOAD STEP= 1	ITERATION= 10	CUM. ITER
NOTE - HEAT FLOW RATES ARE POSITIVE INTO THE NODE	annum persidental sandri di Sandri di Sandri di La Laginani a siddi di Sandri di Sa	
NCDE HEAT		
85 -0.970876E-02 86 -0.194175E-01 87 -0.194175E-01		
88 -0.194175E-01 89 -0.194175E-01 90 -0.194175E-01 91 -0.970876E-02		THE TOTAL AND
TOTAL -0.116505 *** LOAD SIEP 1 ITER 10 COMPLETED TIME 0.000000E+CO KUSE 0 STEADY STATE CONVERGENCE VALUE = 0.5384 AT NODE 22 CRITERION =	CUM. ITEH.= 1.0000	4
***** INPUT FILE SWITCHED FROM FILE27		
PRINTOUT RESUMED BY /GCP		I le.
PRINTOUT RESUMED BY 760P		
**** ROUTINE COMPLETED **** CP = 65.440		
/FOF ENCOUNTERED ON FILE18	The property of the second section of the secti	
	Factories = # 91	

TIMF= 10.1360

65,4800

**** RUN COMPLETED ****

CP=

VAX/VMS ATONY SKST4L 8-MAY-19 VAX/VMS ATONY SKST4L 8-MAY-19 VAX/VMS ATONY SKST4L 8-MAY-19	84 10:10 LFA0: 8-MAY-1	984 10:22 984 10:22 984 10:22	USER3: (ANSYS.TONY) SNST41 USER3: (ANSYS.1UNY) SNST41 USER3: (ANSYS.TUNY) SNST41	L.0UI:11/
AAA 11111 A A I A A I A A I AAAA T A A I A A T	000			
\$	1		••••	The second secon
000000 UU UUUUUUUUU TI	;;;; 11 ;;;; 1111 ;;;; 1111 ;;;; 111 ;;;; 11 ;;;; 11 ;;;;; 11 ;;;;;; 11 ;;;;;;;;	11 111 1111 1111 1111 111 111 111 111	77777771 77771777 77 77 77 77 77 77 77 7	
AAA 11111 A A T A A T A A A T A A A T A A A T	OOO N N Y Y OOO N N N Y			
VAX/VMS ATONY SNST4L 8-MAY-19 VAX/VMS ATONY SNST4L 8-MAY-19 VAX/VMS ATONY SNST4L 8-MAY-19	84 10:10 LPA0: 8-MAY-1		USER3: [ANSYS.TUNY] SNST41 USER3: [ANSYS.TUNY] SNST41 USER3: [ANSYS.TUNY] SNST41	L.OUI;11

```
********* ANSYS INPUT DATA

6 12 18 24 30 36 42 48 54

/PREPT
/TITLE INHUST CHAMBER MODEL
/GOPR
KAN.O

TOFFST,0

EX.1.42.1.2
EX.1.685E6..002E6
EX.3.17E
EX.1.685E6..0012E6
ALPX:13.7FE6..0012E-6
ALPX:13.7FE6..0012E-6
ALPX:13.7FE6..0012E-6
ALPX:13.7FE6..0012E-6
ALPX:13.7FE6..0012E-6
ALPX:3.7.529E-6.4.383E-9.1.007E-11.-7.202F-15.1.786E-18
DENS.2.1
DENS.2.1
DENS.2.1
DENS.2.1
DENS.2.1
NUXY.1.23
NUXY.1.23
NUXY.1.23
NUXY.1.23
NUXY.1.3.635
N. NO.1.1.2045.0.09
FILL
N.57.1.3085E56.009
N.50.1.30388E9.00
N.50.1.30388E9.00
N.50.1.30388E9.009
N.50.1.30388E9.009
N.50.1.3038556.0-
FILL
N.77.1.3202222.009
N.77.1.
                                                                                                                                                                                                                                                                                                                                                                                                           ANSYS INPUT DATA LISTING (FILE18)
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                               78
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                              66
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                      72
                           12
13
14
15
                           16
17
18
                           19012345
                         41
                           44
                           46
                           48
                                                       MAT,3

E,44.51.50,43

EGEN.6.1.37

EGEN.6.7.37,42

ITER.-10.10

KTEMP.1.10

KAC.1

CPSIZE.20

CPNGEN.1.UY.1.85.7

AFWRIT.1.1

FINISH

/EOF
51
-52
-53
-54
-55
   5789012345
                         66
                           68
```

```
authorities to the order voluments that the contracted
  PROPERTY TABLE.DENS MATE 1 NUM. PUTNISE 2
, TEMPERATURE DATA TEMPERATURE
0.0000000E+00 1.000000 2300.000
                                                                  DATA
   MATERIAL 2
CO = 1_000000
                               COEFFICIENTS OF DENS VS. TEMP FOUATION
PROPERTY TABLE, DENS MAT= 2 NUM POINTS= 2 DATA
                         1.000000
                                                             1.000000
        0-0000000E+00
                                             2300.000
   MATERIAL 3
CO = 1.000000
                               COEFFICIENTS OF DENS VS. TEMP FRUATION
 COEFFICIENTS OF NUXY VS. TEMP FQUATION
   CO = 0.2300000
  PROPERTY TABLE, NUXY MAT= 1 NUM. POINTS= 2
TEMPERATURE DATA TEMPERATURE DATA
0.000000E+00 0.2300000 2300.000 0.2300000
   MATERIAL 2
CO = 0.2300000
                               COEFFICIENTS OF NUXY VS. TEMP FRUATION
PROPERTY TABLE, NUXY MATE 2 NUM POINTS 2
, TEMPERATURE DATA TEMPERATURE DATA
0.0000000E+00 0.2300000 2300.000 0.2300000
  MATERIAL 3
                              COEFFICIENTS OF NUXY VS. TEMP FRUATION
    CO = 0.3550000
PROPERTY TABLE.NUXY MATE 3 NUM PUTNTS= 2
TEMPERATURE DATA TEMPERATURE DATA
0.00000000E+00 0.3550000 2300_000 0.355000
                                                            0.3550000
NODE
        1 KCS= 0 X.Y.Z= 1.2945 0.90000E-02 0.00000E+00
         7 KCS= 0 x.Y.Z= 1.2945
                                                  0.00000E+00 0.00000E+00
 NODE
FILL 5 PUINTS BETWEEN NODE 1 AND NODE START WITH NODE 2 AND INCREMENT BY 1
GENERATE 4 TOTAL SETS OF NODES WITH INCREMENT 7
SET IS FROM 1 TO 7 IN STEPS OF 1
GEOMETRY INCREMENTS ARE 0.13333F-02 0.00000E+00 0.00000E+00
GENERATE 4 TOTAL SETS OF NODES WITH INCREMENT 7
SET IS FROM 22 TO 28 IN STEPS OF 1
GEOMETRY INCREMENTS ARE 0.50000F=03 0.00000F+00 0.00000E+00
  NODE
          50 KCS= 0 X,Y,Z= 1.3039
                                                  0.90000E-02 0.00000E+00
NODE
        56 KCS= 0 X,Y,Z= 1-3039 0.00000E+00 0.00000E+00
  TILL 5 POINTS BETWEEN NODE 50 AND NODE
START WITH NODE 51 AND INCREMENT BY
           57 KCS= 0 X.Y.Z= 1.3086
                                                  0.90000E-02 0.00000E+00
  NODE
                                                  0.00000E+00 0.00000E+00
  NODE
           63 KCS= 0 X.Y.Z= 1.3086
  FILL 5 POINTS BETWEEN NODE 57 AND NODE START WITH NODE 58 AND INCREMENT BY 1
NODE
          64 KCS= 0 X.Y.Z= 1.3140
                                                0.00000E+00 0.00000E+00
  NODE
           70 KCS= 0 X,Y,Z= 1.3140
  TIL 5 POINTS BETWEEN NODE 64 AND NODE START WITH NODE 65 AND INCREMENT BY 1
                                                  0.90000E-02 0.00000E+00
  NODE
              KCS= 0 X.Y.Z= 1.3202
           77 KCS= 0 X.Y.Z= 1.3202
                                                  0-00000E+00 0-00000E+00
  NODE
 FILL 5 POINTS BETWEEN NODE 71 AND NODE START WITH NODE 72 AND INCREMENT BY
```

```
TITLE
                                                                                                                                   10.1389
                **** ANSYS ANALYSIS DEFINITION (PREP7) ****
 NEW TITLE THRUST CHAMBER MODEL
 PRINTOUT RESUMED BY /GOP
  ANALYSIS TYPE= 0
 TEMPERATURE OFFSET FROM ABSOLUTE ZERO= 0.000
  REFERENCE TEMPERATURE= 70.000 (TUNIF= 70.000)
ELEMENT TYPE 1 USES STIF42

KEYOPT(1-9)= 0 0 1 2 0 0 0 0 INOTPR= 0 NUMBER OF NODES= 4
  ISOPAR. STRESS SOLID. 2-D
 CURRENT NODAL OUF SET IS UX UY
  TWO-DIMENSIONAL STRUCTURE
                                                                       VS_ TEMP EQUATION
C2 = 0.0000000E+00
    PROPERTY TABLE, EX NATE 1 NUM POINTS 2
TEMPERATURE DATA TEMPERATURE
0.000000000+00 6600000. 2300.000
                                                                               DATA
                                                                              50000000
    CO = 0.1825000E+08 C1 = 1200.000
C3 = 0.0000000E+00 C4 = 0.0000000E+00
                                                                       MATERIAL
PROPERTY TABLE, EX
TEMPERATURE
         RTY TABLE, EX MAT= 2 NUM. POINTS= 2
TEMPERATURE DATA TEMPERATURE DATA
0.0000000E+00 0.1825000E+08 2300.000 0.2101000E+08
                           COEFFICIENTS OF EX
   MATERIAL 3
C0 = 0.1700000E+08
                                                                       VS. TEMP EQUATION
 0.1700000E+08
   0.000000E+00
 PRUPERTY TABLE, ALPX MATE 1 NUM. POINTS= 2
TEMPERATURE DATA TEMPERATURE
0.00000000E+00 0.3970000E-05 2300.000
                                                                            0.6730000E=05
   MATERIAL 2 COEFFICIENTS OF ALPX VS. TENP EQUATION CO = 0.7529000E-05 CI = 0.4383000E-05 C2 = 0.1007000E-10 C3 = -0.7202000E-14 C4 = 0.1786000E-17
         TEMPERATURE DATA TEMPERATURE

0.0000000E+00 0.7529000E-05 100.0000
200.0000 0.8753642E-05 300.0000
400.0000 0.1047819E-04 500.0000
600.0000 0.1245983F-04 700.0000
800.0000 0.1452432E-04 900.0000
1000.000 0.16566005
 PROPERTY TABLE ALPX MATE
TEMPERATURE DA
                                                                            DATA
                                                                            0.8060977E-05
0.9570213E-05
0.1144938E-04
0.1348993E-04
0.1555194E-04
                               0.7529000E-05 100.0000

0.8753642E-05 300.0000

0.1047819E-04 500.0000

0.1245983F-04 700.0000

0.1452432E-04 900.0000

0.1656600E-04 1100.000

0.1854779E-04 1500.000

0.2050171E-04 1700.000

0.2252634E-04 1700.000

0.2479185E-04 1900.000

0.2753500E-04 2100.000

0.3106163E-04 2300.000
                                                                            0.1952340E-04
0.2149588E-04
0.2361582E-04
           1200.000
           1800_000
2000_000
2000_000
   MATERIAL 3
CO = 0.9800000E-05
                                      COEFFICIENTS OF ALPX VS. TEMP FOUATION
 PROPERTY TABLE ALPX MAT= 3 NUM_ POINTS= 2
, TEMPERATURE DATA TEMPERATURE
0_000000000+00_0_98000000E-05_2300_000
                                                                           0.9800000E=05
```

SWANSON ANALYSIS SYSTEMS, INC. HOUSTON, PENNSYLVANIA 15342 PHONE (412)746-3304 TW

TWX 5:

5/ 8/84 (

```
NODE
           78 KCS= 0 X,Y,Z= 1.3272
                                                      0.90000E-02 0.00000E+00
  NODE
           84 KCS= 0 X,Y,Z= 1.3272
                                                      0.00000E+00 0.00000E+00
   TLL 5 POINTS BETWEEN NODE 78 AND NODE
START WITH NODE 79 AND INCREMENT BY 1
  FILL
           85 KCS= 0 X.Y.Z= 1.3350
                                                      0.90000E-02 0.00000E+00
  NODE
  NODE
           91 KCS= 0 X.Y.Z= 1.3350
                                                     <u>0,00000E+00 0,00000E+00</u>
  TILL 5 POINTS BETWEEN NODE 85 AND NODE
START WITH NODE 86 AND INCREMENT BY 1
 MATERIAL NUMBER SET TO 1
FLEMENT 1 2 9
  FI. EMENT
GENERATE 6 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF 1
SET IS 1 TO 1 IN STEPS OF 1
NUMBER OF ELEMENTS= 6
GENERATE 3 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF .7
SET IS 1 TO 6 IN STEPS OF 1
NUMBER OF ELEMENTS= 18
MATERIAL NUMBER SET TO 2
FLEMENT 19 23 30 29 22
 RENERATE 6 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF SET IS 19 TO 19 IN STEPS OF 1
NUMBER OF ELEMENTS 24
  GENERATE 3 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF 7 SET IS 19 TO 24 IN STEPS OF 1 NUMBER OF ELEMENTS 36
  MATERIAL NUMBER SET TO 3
FLEMENT 37 44 51 50 43
  GENERATE 6 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF SET IS 37 TO 37 IN STEPS OF 1
NUMBER OF ELEMENTS= 42
 GENERATE 6 TOTAL SETS OF ELEMENTS WITH NODE INCREMENT OF SET IS 37 TO 42 IN STEPS OF 1 NUMBER OF ELEMENTS: 72

NITTER: -10 NPRINT: 10 NPOST: 10 NOTICER OUNDARY CONDITIONS STEPPED DUE TO NEGATIVE NITTER
  ALL PRINT CONTROLS RESET TO 10
  ALL POST DATA FILE CONTROLS RESET TO 10
 KTEMP= 1 10
  TEMPERATURES FROM LUAD STEP= 1 ITERATION=
STEP BOUNDARY CONDITION KEY= 1
  REACTION FORCE KEY= 1
SPECIFIED DISP UY FROM NODE 7 TO NODE 91 TN STEPS OF 7 VALUES= 0.00000E+00 0.00000E+00 ADDITIONAL DOFS=
  SFT MAXIMUM COUPLED NODE SET SIZE TO 20
  GENERATE ADDITIONAL NODES ON SET= 1 DIRECTION= UY STARTING NODE= 1 ENDING NODE= 85 NODE INC.=
                T= 1 DIRECTION= UY TOTAL NODES= 13
1 8 15 22 29 36 43 50
71 78 85
  NODES=
WAXIMUM COUPLED SET NUMBER= I
  DATA CHECKED - NO FATAL ERRORS FOUND. CHECK OUTPUT FOR PCSSIBLE WARNING MESSAGES.
  ANALYSIS DATA WRITTEN ON FILEZT ( 246 LINFS)
 ALL CURRENT PREPT DATA WRITTEN TO FILE 16
FOR POSSIBLE RESUME FROM THIS POINT
```

	***** NOTICE **** TNITE ELEMENT COMPUTE SYSTEMS. INC. NOR THE FACILITIES FOR THIS INC. THE VALIDITY. ACCURATE DRIVER OF THE ANS DWN RESULTS.	TER PROGRAM. E CORPORATION ANALYSIS ASSU CY. OR APPLIC	NETTHER SWAN SUPPLYING TH ME ANY RESPON ABILITY OF AN	SON ANALYSI E COMPUTER SIBILITY FO Y RESULTS	R	
	SWANSON ANALYSIS SYS ANSYS PROGRAM AS COM POSSIBLE SUGGESTION ERRORS ENCCUNTERED IN RESULTS SHOULD BE IM	PLETE, ACCURA NS AND COMMEN N EITHER THE	TE. AND FASY TS ARE WELCOM DUCUMENTATION	TO USE AS ED. ANY OR THE		
	**** ANALYSIS OPTICE	NS ****				
		VALUF			,1	
ANALYSIS T	YPE	0				
FLEMENT COI REACTION FO MASTER DOF	GREES OF FREEDOM KEY NSTANT TABLE READ KEY READ KEY ABLE ENTRIES	1 6 1 1 24				
REFERENCE UNIFORM TEN	TEMPERATURE	70.00 70.00	1. AND THE RESERVE OF			National of the state of the
water the contract of the cont	**** ELEMENT TYPES	****				
	DESCRIPTION		1 2 3	PTIONS 6		HTONI LM
1 42	ISOPAR. STRESS SOL	10. 5-0	0 0 1	S 0 0	0 0 0	0
NUMBER OF EL	EMENT TYPES= 1					
NO'.	**** TABLE OF ELEMP	ENT REAL CONS	TANTS ****			
NUMBER OF RE	EAL CONSTANT SETS=	0		•		

**** ELEMENT DEFINITIONS ****

FLEMENT	NODES		MAT	TYPE	CLASS	ELEMENT REAL CONSTANTS
3 (2 9 8 3 10 9 4 11 10	1 2 3	1 1 1	1 1	0	
5 6	5 12 11 6 13 12 7 14 13 9 16 15	4 5 6 8			0	an and an
8 10 9 1 10 12	0 17 16 1 18 17 2 19 18	9 10 11	<u>î</u>		0	
11 12 13	4 21 20 6 23 22	12 13 15	<u>1</u>	1	0 0 0	
14 1 15 1 16 1	8 25 24 9 26 25	16 17 18	1		0	
17 20 18 2 19 2 20 2	1 28 27 3 30 29	23 23	1 2 2	1	0	
15 15 15 55 26 55	5 32 31 6 33 32 7 34 33	24 25 26	5 5	1 1	0	
24 26 25 36 26 3 27 36	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	27 30	5 5 5	1	0 0 0	The second secon
78 31 79 31 30 31	$ \begin{array}{ccccccccccccccccccccccccccccccccc$	31 32 33 34		· 1	0 0 0	
31 31 32 31 33 31	7 44 43 6 45 44 9 46 45	36 37 38	5 5	1 1	0 0 0	
34 40 35 4 36 42 37 46	1 4H 47	39 40 41 43		1 1 1	0 0 0	
38 45 39 46 40 41	5 52 51 6 53 52 7 54 53	4 4 4 5 4 6	13 13 3	† 1 1	0 0 0	
41 48 42 49 43 5 44 5	9 56 55 1 58 57	47 48 50 51	3 3 3	1 1	0 0 0	
45 5. 46 5. 47 5!	3 60 59 4 61 60 5 62 61	52 53 54	3 3 3	i	0	
48 50 49 50 50 59	8 65 64 9 66 65	55 57 58	3 3 3		0	
51 66 52 6 53 66 54 6	1 68 67 2 69 68	59 60 61	- 3 3 3	<u> </u>	. 0	
55 65 56 66	5 72 71 6 73 72 7 74 73	62 64 65 66 67	$\frac{3}{3}$	1	0	
58 66 59 69 60 70	9 76 75 0 77 76	68 69	3 3	1 1	0 0 0	
61 7 62 7 63 7 64 7 65 7 66 7	3 80 79 4 81 80 5 82 81	72 73 74	98. 88	1 1 1	0	·
67 7	9 86 85	712 73 74 75 76 78 79 80 81 81 82 83	3 3 3	1	0	
68 89 69 8 70 8 71 8	1 88 87 2 89 88	81 82	<u>3</u> 3		0 0	
71 87 72 87	4 91 90	<u>83</u>		<u> </u>	ŏ	THE THE SECOND PROPERTY OF THE SECOND PROPERT

TNTEGPR STORAGE REQUIREMENTS FOR ELEMFNT TOPUT

MEMORY I= 1001 MEMORY II= 182 TOTAL= 1183 MEMORY AVAILABLE= 340000

MAXIMUM NODE NUMBER FOR AVAILABLE AUXILIARY MEMORY STZF= 189498

NUMBER OF ELEMENTS =

72

MAXIMUM NODE NUMBER USED =

91

			nover the the distance of
NODE	(OK R)	(OR THETA)	THXY (OR RT)
•		0 - 90000F - 08	
>	1.2945	0.75000F-0	
3 4	1.2945	0.60000E-07	
5	1.2945	0.30000F-02	
6 7	1.2945	0.15000E-02	
8	1.2945 1.2958	0.00000E+00 0.90000E+07	
10	1.2958	0.75000E-02	
11	1.2958	0.45000F-07	0_00000F+00
12 13	1.2958 1.2958	0.30000E-02 0.15000E-02	
1 4	1.2958	0.00000E+00	0,00000F+00
15 16	1.2972	0.90000E-02 0.75000E-02	
17	1.2972	0.60000F-07	0_00000F+00
1 A 1 9	1.2972	0.45000F-07	
20	1.2972	0.15000E-02	0.00000E+00
71	1.2972	0.00000F+00 0.90000F-07	
23	1.2985	0.75000E-02	> 0_00000F+00
24 25	1.2985 1.2985	0.60000F=02 0.45000F=02	
26	1.2985	0.30000F-02	0_00000F+00
27 28	1.2985 1.2985	0.15000F-02 0.00000F+00	
29	1.2990	0.90000E-02	0-0000F+00
30 31	1.2990	0.75000F-02 0.60000E-02	
32	1.2990	0.45000F-62	0_00000F+00
33	1.2990	0.30000E=02	
35	1.2990	0.00000E+00	0.000000+00
36 37	1.2995 1.2995	0.90000E=07 0.75000E=07	
3 A	1.2995	0.60000F-02 0.45000F-02	
40	1.2995	0.30000F-0	P 6_00000E+00
41	1.2995	0.15000F-07	
43	1.3000	0.90000E-02	0_00000E+00
4.4 4.5	1.3000	0.75000F-07 0.60000E-07	
46	1.3000	0.45000E-0a	0_00000F+00
47 48	1.3000 1.3000	0.30000E=02	
49	1.3000	0.00000F+00 0.90000F+00	
. 51	1.3039	0.75000E-0.2	
52 53	1.3039	0.60000E-07	
54	1.5059	0.30000E-0	0_000000F+00
55 56	1.3039 1.3039	0.15000E=02 0.00000E+00	
57	1 - 3086	0.90000E-08	0_00000F+00
5 9 5 9	1.3086	0.75000E-02	> 0,00000F+00
60 61		0.45000E=02	0 00000F+00 0 00000F+00
63	1.3086 1.3086 1.3086	0.15000E-0a	0,00000F+00
63 64	1.3086 1.3140	0.00000E+00	
65	1.3140	0.75000F-02	> 0,00000F+00
66 67	1.3140	0.60000E-07	0,00000F+00 0,00000F+00
68 69	1.3140	0.30000E=02	0,00000F+00 0,00000F+00
77	1.3140 1.3140 1.3140	0.00000F+00	7 - 0 0 0 0 0 0 F + 0 0 -
7 · 1		0.90000F~0 2	manuscraphing of the state of the second
72	1.3202	0.75000E-02	
73 74	1.3202 1.3202 1.3202	0.60000E-02 0.45000E-02	
75	1.3202	0.30000E-02	0,00000F+00
76 77	1 (202	0.15000E-02 0.00000E+00	
78	1.3272	0.90000E-02	0,00000F+00
79 80	1.3272	0.75000E-07 0.60000E-07	
81 82	1.3272	0.45000E-02	0'-00000F+00
83	1.3272	0.15000E-08	0_00000F+00
84 85	1.3272	0.00000E+00	0,00000F+00
86	1.3350	0.75000E-02	0,00000F+00
87 88	1.3350 1.3350	0.60000E-08	0_00000F+00
89	1.3350	0.30000E-08	0_0000000000000000000000000000000000000
90 91	1.3350 1.3350	0.15000E-02 0.00000E+00	0.00000F+00 0.00000F+00

XMIN= 1-295 XMAX= 1.335 YMIN= 0.0000F+00 YMAX= 0.9000F+02 ZMIN= 0.0000F+00 ZMAX=

TNTEGER STORAGE REQUIREMENTS FOR NODE INPUT

MEMORY I= 0 MEMORY II= 546 MEMORY AVAILABLE 340000

MAXIMUM NODE NUMBER FOR AVAILABLE AUXILIARY MEMORY STZE= 113333

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SWANSON ANALYSIS RUST CHAMBER MUDEL	NGINEEPING AMALYSIS SYS SYSTEMS, INC. H	iém REVI OUSTON. F	ISTUN 4.0 EP PENUSYLVANIA 1		TURE 1015 AU 		
**** MAT	ERIAL PROPERTIES ****	· · · · · · · · · · · · · · · · · · ·		and the control of th	1		and the second s
'MP EX	(LINFAR INTERPOLATION) TEMP EX	TEMP	ΕX	TEMP	EX	TEMP	ΕX
0.0 0.66000E+07	2300.0 0.20000F+07	and the second s	e e e e e e e e e e e e e e e e e e e			MAN on Production and App 2000 1, including 1979 1	eride i abbil. Crammin.ueri — A dependi A
MP EX	(LINEAR INTERPOLATION) TEMP EX	TEMP	EX	[EMP	EΧ	TEMP	ΕX
0.0 0.18250E+08 TERIAL 3	2300.0 0.21010F+08			e delicates, page as pages — g + 11 mass en aelect		THE STREET, AS A PROPERTY OF STREET, S	antingritaine e rainbangi i Angelyapan bingshapa (ne pa
X PROPERTY TABLE	(LINEAR INTERPOLATION) TEMP FX	TFMP			ΕX		EX
0_0 0_17000E+08	2300.0 0.17000F+08			THE PROPERTY OF THE PROPERTY O	en and an annual section of the sect	1 - May 1 - M - M - 1 - 1 - 1 - 1 - 1 - 1 - 1 -	en e
LPX PROPERTY TABLE	(LINEAR INTERPOLATION)	TEMP	ALPX	TEMP	ALPX	TEMP	ALPX
0_0 0_39700E-05	2300.0 0.67300F-05			*****			1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1. 1
	(LINEAR INTERPOLATION) TEMP ALPX	TEMP	ALPX	TEMP	ALPX	TEMP	ALPX
0.0 0.75290E-05 500.0 0.11449E-04 000.0 0.10566E-04 500.0 0.21496E-04	100.0 0.80610F-05 600.0 0.12460F-04 1100.0 0.17564F-04 1600.0 0.22526F-04 2100.0 0.29179F-04	200-0 700-0 1200-0 1700-0 2200-0	0,87536F-05 0,13490E-04 0,18548E-04 0,23616E-04 0,31062E-04	300-0 800-0 1300-0 1800-0 2300-0	0.95702E-05 0.14524E-04 0.19523E-04 0.24792E-04 0.33233E-04	400.0 900.0 1400.0 1900.0	0.10478E- 0.15552E- 0.20501E- 0.26086E-
TERIAL 3							
MP ALPX	(LINEAR INTERPOLATION) TEMP ALPX	TFMP	ALPX	TEMP	ALPX	TEMP	ALPX
0.0 0.98000E-05	2300.0 0.98000F-05			en allemanne (Laure graphicus - valles de la Herrie Laure	and the second s		THE PARTY OF THE P
IENS PROPERTY TABLE	[28] [1] [1] [1] [2] [2] [2] [2] [2] [2] [2] [2] [2] [2	TFMP	DENS	TEMP	DENS	TEMP	DENS
0_0 1_0000 TERJAL 2	2300.0 1,0000	144-44-44-44-44-4-4-4-4-4-4-4-4-4-4-4-4				and and the engine of the engi	
DENS 010 1.0000	(LINEAR INTERPOLATION) TEMP DENS	TEMP	DENS	ТЕМР	DENS	ТЕМР	DENS
TERIAL 3		CONTINUES SANDONE STATE OF THE SANDON PORT	April 1 and	MANAGEMENT OF STREET STREET, STREET STREET, STREET, STREET, STREET, STREET, STREET, STREET, STREET, STREET, ST			
MP DENS	(LINEAR INTERPOLATION) TEMP DENS 2300.0 1.0000	TFMP	DENS	TEMP	DENS	TEMP	DENS

MATERIAL 1

NUXY PROPERTY TAE	BLE (LINEAR I	NTERPOLATION)					
TEMP NUXY	TEMP	NUXY	TFMP	NUXY	TEMP	NUXY		TEMP
0.0 0.23000	2300-0	0.23000						
MATERIAL 2					*			
NUXY PROPERTY TAE TEMP NUXY	BLE (LINEAR I TEMP	NTERPOLATION: NUXY	TEMP	NUXY	TEMP	Nuxy		TEMP
0.0 0.23000	5300.0	0.23000		Personal of English Company of the Professional	gara di manani na matan na dananakan di mananagan ya ina nyanda da manana	THE THE REPUBLISH AND THE COMMENTAL STREET AND ADMINISTRATION OF THE PERSON OF THE PER		arigania areng pangganian, nga a pangganian sa mananan
MATERIAL 3		•						
NUXY PROPERTY TAG TEMP NUXY	OLE (LINEAR I TEMP	NTERPOLATION: NUXY	TFMP	NUXY	ТЕМР	NUXY		TEMP
0.0 0.35500	5300.0	0.35500			Makes a seminatahan menangan seminan di semi	THE THE PLANT OF THE PARTY OF T	***	and the second second second second second
MAXIMUM MATERIAL NU	MBER= 3					. ,		
****	COUPLED DEG.	OF FR. DEFINI			THE PERSON NAMED IN CONTRACTOR OF THE PE	With the second state of the second s		
SFT 0.0.F.	NUMBER	COUPLED 1	400FS					
1 UY	1 3	1 A 71 78	15 85	•	29 36	43	50	57
NUMBER OF COUPLED	BETS= 1	a recommendation of the second second second second second						
NOOF DEGREE NUMBER OF SPECIFIED TOTAL NUMBER OF MAS								
TNTEGER STORAGE REMEMORY I= 2172	GUIREMENTS	OR MATERIALS.	FTC. IN	PUT 2 MEMORY	CP= AVAILABLE=	17.870 T	IMF=	10.14935
*** LOAD STEP 1	OPTIONS SPEC	IFICATIONS						
NTTTER= -10 NPF		NPOST= 10						
ALL PRINT CONTROLS	RESET IU	10						
ALL POST DATA FILE	CONTROLS RES	ET TO 10						
ACEL= 0.00000E+00	.00000E+00 0	.00000E+00						
OMEGA= 0.00000E+00	0.00000E+00	0.00000F+00						
DOMEGA= 0.00000E+00	0.00000E+00	0.00000F+00						
CGLOC= 0.00000E+00	0.00000E+00	0.00000F+00						
CGDMEGA= 0.00000E+6	0.00000E+0	0 0'00000F+0)			of the Automorphism of the		
DOMEGA= 0.00000E+00			21120					
KTEMP= 1 10			The control and the second					
TEMPERATURES FROM L	OAD STEP=	1 ITFRATTO			and the second s			
PLASTICITY CONVERGE CREEP OPTIMIZATION LARGE DEFL. CONVERGE DISPLACEMENT LIMIT KEY TO TERMINATE RI	: 0_00000F+00	A- U.UUIU			**			
REACTION FORCE KEY	: 1							
INTEGRM TEMPERATURE	= 70.000	(TREF= 70.6	100)			4		

SWANSON ANALYSIS	GIN eering analys is- Sysiems. Inc.	SYSTFM RFV HOUSTON.	ISTON 4.0 EZ PENNSYLVANIA	15342 PI	TURBINES AUG	1,198 04 TI
THRUST CHAMBER MODEL		•			10-1499	5/ 8/
I DAD STEP NUMBER = 1						
*** LOAD OPTIONS SUMM	ARY					
TIME = 0.00000E+00 NIT	TER= -10		and the same of th	pagamagan , hinis milihidiki V ^A Tanggi nganggangan e v e sel e s	onstandaren auga en fortretar en en en en e n en	
ACEL = 0.000000E+00 0. OMEGA = 0.00000E+00 0. DOMEGA= 0.00000E+00 0. CGLOC = 0.00000E+00 0. CGOMEG= 0.00000E+00 0. DCGOMEG= 0.00000E+00 0. TFMPERATURE (KTEMP) LO STIFFNESS REUSE KEY (K	00000E+00 0.00000F+ 00000E+00 0.00000E+ 00000E+00 0.00000E+ 00000E+00 0.0000E+	00 00 00 00				
IINTFORM TEMPERATURE=	70.000 (TREE= 7	0_000)	•			
PRINT FLEMENT FORCES A					A SECTION OF THE SECT	
I DADS STEPPED TO FINAL PLASTIC CONVERG. CRITE CREEP OPTIMUM. CRITERI LARGE DEFL. CUNVERG. COTSPLACEMENT LIMIT. O. KFY TO TERMINATE RUN I MODE:	R1!ERIUN= 0.0010	0.0				er e are
NPRINI= 10 NPOST= DISP_ POST DATA FREG=	10 REACTION P 10 REACT. POST	RINT FREQ= DATA FREQ=	1 0 : 1 0			
DISPLACEMENT PRINT FR FREG NSTRT NSTOP 10 1 32000						
FLEMENT PRINT AND PCS TYPE STIFF STRESS NO. PRINT 1 42 10	FORCE STRESS D PRINT DATA L	ATA FORCE EVEL DATA 3 10				
NODF UX	FIED DTSPUACEMENTS UY	****				
7 14 21 26 35 42 49 49 56 63 70 77 84	0.000000F+00 0.000000F+00 0.000000E+00 0.000000E+00 0.000000E+00 0.000000E+00 0.00000E+00 0.00000E+00 0.00000E+00 0.000000E+00 0.000000E+00					
TEMPERATURE DATA READ	FROM LOAD STEP=	1 ITFR=	10 CUM. I	TER= 4	ON FILE 4	
**** LOAD SUMMARY -	13 DISPLACEMENTS	() F(neces o	PRESSURFS	****	
INTEGER STORAGE REGUL MEMORY I= 1316 MEM	REMENTS FOR LUAD DA ORY II= 0 TO	TA INPUT	6 MEMORY AV	CP= P AILABLE=	3.430 FIME= 10. 340000	15279
**** CENTR	OID. MASS. AND MASS	MUMENTS OF	INFRTIA ***	**		
CALCULATIONS ASSUME E				• •		The same of the sa
TOTAL MASS = 0.50111	E=02					<u></u> -

```
MASS AND MOMEN' VALUES ARE EVALUATED FOR A 360. DECREE MODEL.
 THE BELOW TABLE IS BASED ON A WEIGHTEDIMASS PROPURTIONAL TO RADIUS) AXISYMMETRIC SECTION.
                                                                                  SQUARE ROOT OF SECOND MOMENT/MASS (RADIUS OF GYR.)
                   FIRST FIRST MOMENT/MASS MOMENT (CENTROID)
                                                                   SECOND
                                                                SELD...
MOMENT
              0.3959E-02 1.3149
0.1355E-04 0.45000E-02
 RADIAL
                                                                 0.5206F-02
                                                                                           1.315
   *** MASS SUMMARY BY ELEMENT TYPE ***
          MASS
        0.301107E-02
RANGE OF ELEMENT MAXIMUM STIFFNESS IN GLOBAL COORDINATES WAXIMUM= 0.598372E+08 AT ELEMENT 72 MTNIMUM= 0.133552E+07 AT ELEMENT 3
TRIEGER STORAGE REGLIREMENTS FOR ELEMENT FORMULATION CP= 39.670 TIME= 10.15987

MEMORY I= 1314 MEMORY II= 0 TOTAL= 1314 MEMORY AVAILABLE= 340000
  *** FLEMENT STIFFNESS FORMULATION TIMFS
TYPE NUMBER STIF TOTAL CP AVE CP
        12
                      42
                               15.360
                                            0.213
  TIME AT END OF ELEMENT STIFFNESS FORMULATION CP =
                                                                           39.690
 MAXIMUM IN-CORE WAVE FRONT ALLOWED FOR REQUESTED MEMORY SIZE: 579
  TNTEGER STURAGE REQUIREMENTS FOR MAVE FRONT MATRIX SOLUTION CP= 45.640 TIME= 10.16166 FMORY I= 1314 MEMORY II= 361 TOTAL= 1675 MEMORY AVAILABLE= 340000
TWAXIMUM IN-CORE WAVE FRONT (EQUATIONS) USFD= 17
  *** MATRIX SOLUTION TIMES
READ IN ELEMENT STIFFNESSES
NODAL COORD. TRANSFORMATION
MATRIX TRIANGULARIZATION
 TIME AT END OF MAIRIX TRIANGULARIZATION CHE
                                                                          45.660
  TNTEGER STORAGE REQLIREMENTS FOR HACK SUBSTITUTION CP= 46_400 T1MF= 10_16193 FMORY I= 1314 MEMORY II= 370 TOTAL 1684 MEMORY AVAILABLE 340000
  *** FLEM. STRESS CALC. TIMES
TYPE NUMBER STIF TOTAL CP
                                          AVE CP
   1 72 42 6.230 0.087
   *** NODAL FORCE CALC. TIMES TYPE NUMBER STIF TOTAL CP
                                           AVE CP
   1 72 42 0.340 0.005
  *** LOAD STEP 1
                              ITER
                                          1 COMPLETED. TIME= 0.000000E+00 KUSE= 0 CUM. ITER.=
  *** SOLUTION CONVERGED - LOAD STEP | CONVERGED AFTER ITERATION | 1 CUM. ITER. = 1 NEXT ITERATION (IDENTIFIED AS ITERATION 10) SATISFIES PRINTOUT OR POST DATA REQUEST.
```

** D	ISPLACEMENT SOL UX	UTION ***** T	IMF = 0.00	0000E+00 L	OAD STEP=	1 ITERATION=	
1	0.477796E-02	0.335788E-04		•	,		
2	0-477796E-02	0-279823E-04			•		
3	0.477796E-02 0.477796E-02	0.223859E-04		-			
5	0.477796E-02	0.167894E=04 0.111930E=04					
6	0.477796E-02	0.559650E-05			•		
7	0.477796E-02	0.00000E+00					
3	0-482023E-02	0.335788E-04			*******************************		
₹	0-482023E-02	0-279823E-04					
)	0.482023E=02	0.223859E-04	1.5				
}	0.482023E-02 0.482023E-02	0.167894E-04 0.111929E-04				**************************************	
ţ	0.482023E-02	0.559647E-05					
1	0.482023E-02	0.00000E+00					
;	0.484649E-02	0.335788E-04					
	0-484649E-02	0.279823E=04		and distributions are as a second	- 41 Tel - Contragation Contract to the Contract	The second section and the second section of the second section and the second section	
	0-484649E-02	0.223859E-04					
į	0-484649E-02	0-167894E-04	•				
	0.484649E-02 0.484649E-02	0-111929E-04					
	0.484649E-02	0.559647E-05 0.00000E+00					
	0.4855066-08	0.335788E-04					
	0.485506E-02	0.279823E-04					
	0.485506E-02	.0.223859F-04		***************************************		and the second s	
	0-485506E-02	0.167894E-04					
	0.485506E-02	0-111929E-04					
	0-48550hE-02	0.559647E-05					
	0-485506E-02	0.000000E+00		1.18.4.1.100	return to a a co	The second secon	
	0.485851E-02	0.335788E-04					
	0.485851E=02	0.279823E-04					
	0.485851E-02 0.485851E-02	0.223859E-04	-			The second section of the section	
	0.485851E-02	0.167894E-04 0.111929E-04					
	0.485851E-02	0.559647E-05					
	0.485851E-02	0.00000E+00					
	0.466153E-02	0.3357d8F-04	** ** **			* * * * * * * *	
	0-486153E-02	0.279823E-04					
	0.486153E-02	0.223859E-04					
	0-486153E-02	0.167894E-04					
	0-486153E-02	0.111929E-04				*****	
	0.486153E-02	0-5596476-05					
	0.486153E-02 0.486412E-02	0.000000E+00 0.335788E=04					
	0.4864122-02	0.279823E-04			11	The second of th	
	0.486412E-02 0.486412E-02 0.487950E-02	0.559647E-05 0.00000E+00 0.335788E-04					
	0.487950E-02	0.279823E=04					
	0.487950E-02	0.223859E-04				and the second of the second o	
	0-487950E-02	0-167894E-04					
	0-487950E-02	0.111929E-04	•				
	0-487950E-02 0-467950E-02	0.559647E-05 0.00000E+00		**** *** ******************************			
	0.489717E-02	0.335788E-04					
	0.489717E-02	0.279823E-04					
	0.489717E-02	0.223859F-04					
	0-489717E-02	0.167894E-04					
	0-489717E-02	0-111929E-04			*,		
	0.489717E-02	0.559647E-05	_		<u>.</u>		
	0.489717E-02	0.000000E+00	-				٠
	0.491671E-02 0.491671E-02	0.335788F=04					
	0-491671E-02	0.279823E-04 0.223859E-04					
	0-491b71E-02	0-167894F-04					
	0-491671E-02	0.111929E-04			* ***	The second secon	~
	0-491671E-02	0.559647E-05					
	0.491671E=02	0.000000E+00					
	0-493764E-02	0.335788E-04 0.279823F-04					
	0.493764E-02	0.2238596-04					
	0.493764E=02	0.1678948-04					
	0-493764E-02	0.111929F-04					
	0-493764E-02	0.559647E=05					
	0-493764E-02	0.000000E+00					
	0.495943E+02	0.335788E-04					
	0.495943E-02	0.279823E-04 0.223859E-04					
	0.495943E-02	U-223039F=04 U-167894E=04					
	0.495943E-02	0.111929F-04					
	0.495943E-02	0.559647E-05					
	0.495943E-02		The second secon		***	*****	
	0.498146E-02	0.335788F-04					
	0.498146E=02	0-279823F-04					
	0.498146E-02	0.223859E-04 0.167894E-04			p W		
	244201405-05	V.10/074E#04					
	0-498146E-02	ำก เปลือกกร กร					
	0.498146E-02	0.111929E-04 0.559647E-05					
	0.498146E-02	0.000000E+00					
			-				
UMS	88	•					
	0.498146E-02	0-335788F=04	140				

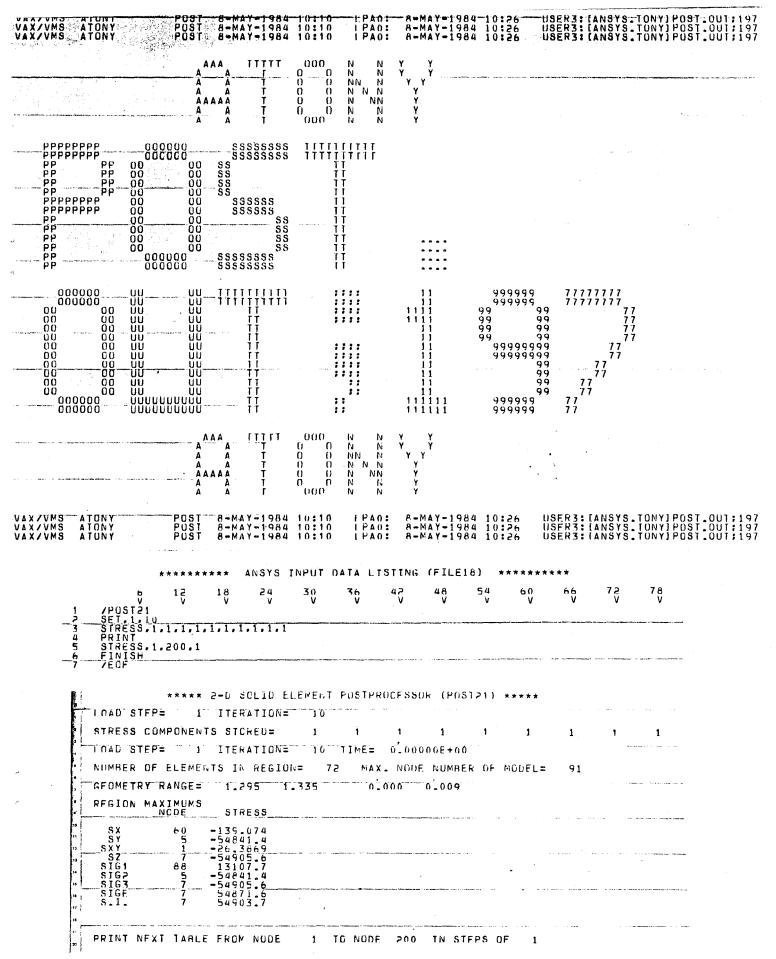
THICST CHAMBER MODEL		10.1648 57 6784 CP= 55.760	
***** ELEMENT STRESSES ***** TIME = 0.000000E+00 TUAL STEP= 1			
TE 1 NODESE 2 9 8 17, MATE 1 VOLE 0.2590F-05 2.7CE 1.295 0.8250E-02 TEMPE3227.6. SX.SY.SXY.SZE -24.701 SIG1.SIG2.SIG3= -24.701 SIG1.FORCES ON NODE 9 0.45526F-06 -41.9627 STATIC FORCES ON NODE 9 0.45526F-01 -36.6321 STATIC FORCES ON NODE 8 0.45559F-01 36.6321 STATIC FORCES ON NODE 1 0.199640F-13 41.9627	-45525. 45539.	-0.11062E-02 -45564. SIGE= 45520.	
T = 7 NGDES= 3 10 9 2	-45525. 45539.	0.72755E-03 -45564. SIGE= 45520.	
### 3 NODES ### 11 10 3 , MAT 1 VOL 0.2590F-05 ####################################	-45525. 45539.	0.39518E-03 -45564. SIGE= 45520.	2+D SCTID
#C= 4 NGDES= 5 12 11 47, MAT= 1 V01= 0.2590E-U5 #C:YC= 1.295 0.3750E-02 TEMP=3227.6 SX.SY.SXY.SZ= -24.701 #IGL:SIG2.SIG3= -24.701 -45525455.44 S.1.= #FABIC FURCES ON NOCE 12 0.45540F-01 -41.9627 #FABIC FURCES ON NCDE 11 0.45540F-01 36.6321 #FABIC FURCES ON NCDE 11 0.45540F-01 36.6321	-45525. 45539.	-6.39518E-03 -45564. SIGE= 45520.	2-D SOLID
TY = 5 NODES = 6 13 12 5 . MAT = 1 VOL = 0.2590E-05 xC_YC = 1.295	-45525. 45539.	-0.727,55E-03 -45564. SIGE= 45520.	2-D SCLID
TE 6 NGDESE 7 14 13 6 , MATE 1 VDL 0.2590E-05 XC.YC 1.295 0.7500E-03 TEMPE3227.6	-45525. 45539.	0.11062E-02 -45564. SIGE= 45520.	2-D SOLID
TT = 7 NODES = 9 16 15 8 , MAT = 1 VOL = 0.2594E-05 *C.YC = 1.297 0.8250E-02 TFMP=2161.9, \$X.\$Y.\$XY.\$7 = -63.463 *SIG1.\$IG2.\$IG3 = -63.463 *SIG1.\$IG = 0.8250E	-29583- 29523:	0.47900E-03 -29586. SIGE= 29521.	2-D SCLID
### ### NODES 10 17 16 9 , MAT 1 VOI 0.2594E-05 ####################################	-29583. 29523.	-0.13724E-02 -29586. SIGE= 29521.	2-D SOLID
TE 9 NODES 11 18 17 10 , MATE 1 VOLE 0.2594E-05 #C.YCE 1.297 0.5250E-02 TEMP=2161.9, \$X.SY.SXY.S7 = -63.462 SIG1.SIG2.SIG3	-29583. 29523.	-0.31594E-03 -29586. SIGE= 29521.	2-D SOLID
TI TO NODES 12 19 18 11 , MATE 1 VOL 0.2594E-05 *C.YC= 1.297 0.3750E-02 TEMP=2161.9 SX.SY.SXY.SZ -63.462 *IG: SIG: SIG: -63.462 -29583 -29586. S.I.= *STATIC FORCES ON NOCE 12 -0.45531E-01 -24.5516 *TITIT FORCES ON NOCE 19 0.751559F-01 -22.6113	-29583. 29523.	0.31594E-03 -29586. SIGE= 29521.	5-D SOLID
STATIC FUNCES UN NOBE 18 0.751564F=01 28.5516 FIE 11 NODES 13 20 19 2 MAIE 1 VOI = 0.2594E=05 SIGI.SIG2.SIG3 = -63.461 -29583 -29586 S.I.= SIGI.SIG2.SIG3 = -63.461 -29583 -29586 S.I.= SIGI.FORCES ON NOBE 13 -0.455503F=01 -28.5516 STATIC FORCES ON NOBE 19 0.751549F=01 -28.5516 STATIC FORCES ON NOBE 19 0.751549F=01 -28.5516	-29583. 29523.	0.13724E-02 -29586. SIGE= 29521.	
TI = 12 NUDES = 14 21 20 13	-29583. 29523.	-0.47900E-03 -29586. SIGE= 29521.	2-0 SOLID
T= 13 NODES= 16 23 22 15 MAT= 1 VOI = 0.2595E-05 (C.YC= 1.298 0.3250E-02 TEMP=1097.3 SX.SY.SY.S7= -84.370 -9716.0 -9743.8 S.I.= SIGILS FORCES ON NODE 23 0.348726F-01 -13.0616 SIGILS FORCES ON NODE 23 0.348726F-01 -3.79159 STATIC FORCES ON NODE 23 0.348726F-01 -3.79159	=9743.8 9659.5	~0.55892E-04 -9716.0 SIGE= 9645.6	2-0 \$CLID
THE FUNCES ON NODE 15 -0.751573F-01 13.0616 THE 14 NODESE 17 24 23 16 C. YCE 1.298 0.6750E-02 TEMPE1097.3 SX.SY.SXY.SZF -084.368 SIGI.SIG2.SIG3= -84.368 -9716.0 -9743.8 S.I.= SIGIT. FORCES ON NODE 17 -0.751564F-01 -13.0616 STATIC FORCES ON NODE 24 0.848710F-01 -3.79159 STATIC FORCES ON NODE 23 0.846717F-01 3.79159 STATIC FORCES ON NODE 16 -0.751552F-01 13.0616	-9743.8 9659.5	-0.56978E-03 -9716.0 SIGE= 9645.6	S÷D SCLID

TATIC FORCES ON NUDE 25 0.848707F=01 -3.79159 TATIC FORCES ON NOCE 24 0.848712F=01 3.79159 TATIC FORCES ON NODE 17 -0.751546F=01 13.0616				
= 16 NODES= 19 26 25 18 NAI= 1 VOL= .YC= 1.298 0.3750E-02 TEMP=1097.3 SX.SY.SXY.SZ= .IGL.SIG3= -84.368 -9716.0 -9743.8 .IATIC FURCES ON NODE 19 -0.751546F-01 -13.79159 .IATIC FURCES ON NODE 26 0.848772F-01 -3.79159 .IATIC FURCES ON NODE 25 0.848772F-01 3.79159 .IATIC FURCES ON NODE 18 -0.751563F-01 13.0616	0.2595E-05 -64.368 5.I.=	-9743-8 9659-5	0.62783E-03 -9716.0 SIGE= 9645.6	2-0 SuLID
TATIC FORCES ON NODE	0.2595E-05 -84.368 S.I.=	-9743.8 9659.5	0.56978E-03 -9716.0 SIGE= 9645.6	e-o sclio
= 18 NGDES= 21 28 27 20 MAT= 1 VGL= :.YC= 1.298 0.7500E-03 TEMP=1097.3 SX.SY.SXY.S7=	0.2595E705	~9743.9	0.55891E-04 -9716.U	e-c sclio
TATIC FURCES ON NOCE 28 0.348729F-01 -3.79159 IFATIC FURCES ON NODE 27 0.848729F-01 3.79159				
TATIC FORCES ON NODE 20 -0.7517/2F-01 13.0616 - T9 NODES= 23 30 29 29	-96.760 -96.760 5.1.=	-48895. 48799.	0.44283E-03 -48763. SIGE= 48732.	
TATIC FORCES ON NUCE 22 -0.848779F-01 16.3867 = 70 NODES= 24 31 30 73 MAT= 2 VOL= 1.97E= 1.299 0.6750E-02 TFMP= 549.1. SX.SY.SXY.SZ= 3IG1.SIG2.SIG3= -96.758 -48763 -48895. 3IATIC FORCES ON NODE 24 -0.848715F-01 -16.3867 3TATIC FORCES ON NODE 31 0.103157 15.3650 3TATIC FORCES ON NODE 30 0.103157 15.3650 STATIC FORCES ON NODE 23 -0.848718F-01 16.3867	0.9741E-06 -96.758 S.I.=	-48895. 48799.	0.68331E-03 -48763. SIGE= 48732.	2-0 SCLID
TATTIC FUNCES ON NODE 23 -0.848718F-01 16.3862 = 21 NODES= 25 32 31 24 MAT= 2 VOI= YC= 1.294 0.5250F-02 TEMP= 549.1 5x.5Y.5XY.5Z= - 91.5192.5163= -96.758 -46763= -4887.5 ATIC FUNCES ON NODE 25 -0.648707F-01 -15.3650 ATIC FUNCES UN NODE 32 0.103157 -15.3650 ATIC FUNCES UN NODE 34 0.103157 -15.3650 ATIC FUNCES ON NODE 24 -0.648708F-01 15.3862	: ŭ.97a16⇒06			2-0 SCLID
ATIC FURCES UN RODE 31 0.103157 15.3650 ATIC FURCES ON NODE 24 -0.448708F-01 16.3862	-9741E-06 -96-758 S-I-=	-48895. 48799.	-0.26698E-03 -48763. SIGE= 48732.	2-0 SGLID 42
ATIC FURCES ON ACDE 33 0.103157 -15.3650 ATIC FURCES ON NODE 32 0.103157 15.3650 ATIC FURCES ON NODE 25 -0.848707F-01 16.3862 73 NODESE 27 34 33 26 MATE 2 VOIE 0 1.299 0.2250E-02 TEMPE 549.1, SX.SY.SXY.SXY.SXZ48763 ATIC FURCES ON NODE 27 -0.848718F-01 -16.3862 ATIC FURCES ON NODE 34 0.103157 -15.3650 ATIC FURCES ON NODE 35 0.103157 -15.3650 ATIC FURCES ON NODE 26 -0.848715F-01 16.3862	1.9741E-06 96.758 5.I.=	*48895. 48799.	-0.68331E-03 -48763. SIGE= 48732.	2-D SOLID 42
ATIC FURCES UN NODE 37 -0.848/18F-01 -16.3867 ATIC FURCES UN NODE 33 0.103157 -15.3650 ATIC FURCES ON NODE 36 -0.848715F-01 16.3862	970:5-06			- 2=0 S0(10 μ2
Pa NODES= 28 35 34 27 MAT= 2 VOL= 6 YC= 1.299				
75 NODES= 30 37 36 29 MAT= 2 VOI = 0 YC= 1.299	19744E-06 112.90 S.I.=	-35477. 35365.	0.52424E-03 -35329. SIGE= 35290.	e÷b sclid 4a
ATIC FORCES ON NOBE 29 -0.103159 12.0281 26 NODES= 31 38 37 30 MAI= 2 VOL= 0 YC= 1.299 0.6750E-02 TEMP= 516.0 SX.SY.SXY.S7= - \$1.SIG2.SIG3= -112.90 -35379, -35477. ATIC FORCES ON NOBE 31 -0.103157 -12.0281 ATIC FORCES ON NOBE 36 0.116406 -11.0188 ATIC FORCES ON NOBE 30 -0.103158 12.0281	5.9744E-06 112.90 5.1.=	-35477. 35365.	0.94951E-03 -35329. SIGE= 35290.	2-0 SOLID 42
ATIC FUNCES UN NOBE 30 -0.103158 12.0281 P7 NODES= 32 39 38 31 MAT= 2 VOL= 0 YC= 1.299 0.5250E-02 TEMP= 516.0 \$X.SY.SXY.S7= - 51.SIG2.SIG3= -112.90 -35329 -35477. ATIC FORCES ON NODE 32 -0.103157 -12.0281 ATIC FORCES ON NODE 36 0.116405 -11.0186 ATIC FORCES UN NODE 36 0.116405 11.0186 ATIC FORCES UN NODE 31 -0.103157 12.0281	1.9744E-06 112.90 S.I.=	-35477. 35365.	0.43595E-03 -35329. SIGE= 35290.	2-D SGLID 42
ATIC FORCES UN NCDE 31 -0.303157 12.0281 78 NODFS= 33 40 39 32 MATE 2 VOLE 0 YC= 1.299 0.3750E-02 TEMP= 516.0 SX.SY.SXY.SYE = - \$1.SIG2.SIG3= -112.90 -35329 -35477. ATIC FORCES ON NODE 33 -0.103157 -12.0281 ATIC FORCES ON NODE 39 0.116406 11.0188 ATIC FORCES ON NODE 32 -0.103157 12.0281	.9744E-06 112.90 S.I.=	-35477 35365.	-0.43596E-03 -35329. SIGE= 35290.	2-D SOLID 48
ATIC FORCES ON NOBE 32 -0.103157 12.0281	.9744F-06 112.90 S.I.=	-35477. 35365.	-0.94951E-03 -35329. SIGE= 35290.	2-D SGLID 4;
30 NODES= 35 42 41 34 , MAT= 2 VOI = 0 YC= 1,299 0.7500E-03 TEMP= 516.0, SX.SY.SXY.SXY.S7= -31.5162.SIG3= -112.90 -3532935477.	.9744E-06 112.90 S.I.=	-35477. 35365.	-0.52424E-03 -35329. SIGE= 35290.	2-D SOLIO 42
ATIC FORCES ON NODE 35 -0.103159 -12.0781 ATIC FORCES ON NODE 42 0.116407 -11.018A ATIC FORCES ON NODE 41 0.116407 11.018A ATIC FORCES ON NODE 34 -0.10315A 12.0281 TO STANDES 37 44 3 36 MATE 2 VOIL 0 YC= 1.300 0.2250E-02 TEMP= 482.6 SX.SY.SXY.S7 = -22588. ATIC FORCES ON NODE 44 0.124806 -7.88739 ATIC FORCES ON NODE 44 0.124806 -6.83127				
AFIE FURCES UN NODE 44 6.124806 -6.83127	151	The state of the s		

TATALLE FUNCES UN NODE	43 0.124806 36 -0.116407	h_H3127 7_82H39	PROF. 171			
FI = 3P NUDES= 38 45 xr. yr = 1.300 0.0750E-0. sig1.sig2.sig3= -123.96 	44 37 2 TEMP= 482.6. -22399- 38 -0-116406	MATE 2 VOI SX.SY,SXY.S7 -2255 -7,82439	1= 0.9748E-06 = -123.96 A. S.I.=	-22558. 22434.	0.91239E-03 -22399. SIGE= 22355.	2=0 SCLID
STATIC FORCES ON NODE STATIC FORCES ON NODE FI = 33 NODES = 39 40 XC.YC= 1.300 0.5250E-0. SIGT.SIG2.3IG3= -123.96	44 0.124806 37 -0.116407 45 38 2 TEMP= 482.6.	7.82839 MAT= 2 VOI SX.SY.SXY.S7	L= 0.9748E-06 = -123.96 8. S.I.=	-22558 - 22434 -	0.43320E-03 -22399. SIGE= 22355.	2-0 SCLID
STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	39 -0.116406 46 0.124806 45 0.124805 38 -0.116406	-7-82839 -6-83127 6-83127 7-82839				
FI = 34 NODES = 40 47 YC. YC = 1.300 0.3750E-0 SIG1.SIG2.SIG3123.96 STATIC FURCES ON NODE STATIC FURCES ON NODE STATIC FURCES ON NODE	2 TEMP= 482.6 40 -0.116406 47 0.124806 46 0.124806	MAT= 2 V0 SX.SY.SXY.S7 -7,82839 -6.83127 6.83127	1 = 0.9748E-06 = -123.96 8. S.I.=	-22558. 22434.	-0.43320E-03 -22399. SIGE= 22355.	2-0 SOCIO
FI = 35 NUDES 41 48 **XC.**C= 1.300 0.2250E-0 **SIG1.**SIG2.**SIG3 = 0.2250E-0 **SIG1.**SIG3.**SIG3 = 0.2250E-0 **SIG1.**SIG3	47 40 2 TEMP= 462.6. -22399- 41 -0.116407 48 0-124806	7-82839 MATE 2 VO SX.SY.SXY.S7 -7.42839 -6-83127	I = 0.9748E-06 = -123.96 RL S.I.=	-22558. 22434.	-0.91239E-03 -22399. SIGE= 22355.	s-n sorio
*C. YEE 1.300 0.7500E=0	3 TEMP= 482 6,	SX.SY.SXY.SZ	I = 0.47486-06 = -123.96 8. S.I.=	-22558. 22434.	-0.48360E-03 -22399. SIGE= 22355.	Z=0 SCLID
STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE TI = 37 NODES= 44 51 xC.YC= 1.302 0.3250E-0 SIGI.SIG2.SIG3= 0.3250E-0 SIGI.SIG2.SIG3= 0.00E STATIC FORCES ON NODE	48 0.124806 41 -0.116406	6.83127 7.82839				e-o scuto
STATIC FORCES ON NOLE STATIC FORCES ON NOCE FI = 16 NODES= 45 52 FC_YC= 11 102 0.6750E-0 SIG1.SIG3= -132.16 STATIC FORCES ON NOCE STATIC FORCES ON NOCE	2 1 44 46 7 1 2 48 0 6 1 2 46 7 1 2 48 0 6 1 2 48 0 5 1 4 4 6 7 1 2 4 8 0 5 2 0 1 1 3 2 3 1 1 5 2 0 0 1 3 3 3 1 1 5 2 0 0 1 3 3 3 1 1 3 2 3 1 3 1	MAT= 3 VO SX.SY.SXY.S7 -A-20214	1.= 0.7595E-05 = -132.16 -5 \$.I.=	-2423.5 -2791.4	0.13731E-03 -2785.1 SIUE= 2724.8	e-o scliv
STATIC FUNCES ON NODE STATIC FUNCES ON NODE TI = 39 NODES = 46 53 CC.YC= 1.302 0.5250E-0 SIGILSIGS.SIGI - 132.16 STATIC FUNCES ON NODE	52 45 2 TEMP= 462 0	MAT= 3 VO SX.SY.SXY.S7	1= 0.7595E+05 = -132.16 -5 Sal.=	6 -2423.5 -2791.4	0.64453t=04 -2785.1 SIGE= 2724.8	e-o scuto
FI = 40 NODFS = 47 54 (C.YC = 1.302 0.3750E-0 SIG1.SIG2.SIG3 = -132.16 STATIC FORCES ON NOCE	53 46 2 TEMP= 462.0 -2785.1 47 -0.124806 54 0.132931	MAT= 3 VO SX.SY.SXY.SZ =2923 =3.20214 =5.59945	1 = 0.7595E-05 = -132.16 -5 S.I.=	-2923.5 -2791.4	-0.64437E-04 -2785.1 STGE= 2724.8	Z=O SGCIO
STATIC FORCES ON NODE FI = 41 NODES= 48 55 (C.YC= 1.302 0.2250E=0 SIG1.SIG2.SIG3= -132.16 STATIC FUNCES ON NODE STATIC FUNCES ON NODE STATIC FUNCES ON NODE	46 -0.124806 54 47 2 TEMP= 462.0 -2785.1 48 -0.124806 54 0.132931	A.20214 MATE 3 VO SX.SY.SXY.SZ -2223 -8.20214 -6.59945	1.= 0.7595E-05 = -132.16 -5 S.1.=	-2923.5 2791.4	-0.13731E-03 -2785.1 SIGE= 2724.6	S-D SOLID
42 42 VODES 49 36	33 46	mar⇒ 3 VII	(= 0 a / 3 7 3 E = 0 3	,		2-D SCUID
### 1.402	0.132931 0.132931 -0.12460p	-n.59945 -59945 8.20214				
43 NODES= 51 58 YC= 1.306 U-8250E-02 G1.81G2.81G3= 1.37.17 ATIC FURCES UN NODE 56 ATIC FURCES UN NODE 57 BT ATIC FURCES UN NODE 57	57 50 7,9 MA TEMP= 453,9 SX -683-40 -0.132931 0.135326 0.135326 -0.132931	T= 3	0.9144E-05 -137.17 S.I.=	-827.00 689.82	-0.19744E-04 -683.40 SIGE= 630.41	ē-D \$GLID 4
YC= 1.306 0.6750E-02 G1.SIG3-137.17 ATIC FURCES ON NODE 59 ATIC FORCES ON NODE 59 ATIC FURCES ON NODE 58 ATIC FURCES ON NODE 58 ATIC FURCES ON NODE 58	58 51 7 MA TEMP= 453.9 SX -643.40 -0.135326 0.135326 0.135326	T= 3 VOI = .SY.SXY.S7= -627.00 -3.67842 -1.36148 1.36148 3.67842	0.9144E-05 -137.17 5.I.=	-827.00 689.82	-0.36647E-04 -683.40 SIGE= 630.41	2-D SCLID 4
YC= 1.306 0.5250E-02 G1.SIG2-SIG3= -137.17 ATIC FORCES UN NUDE 50	59 52 MA TEMP= 453,9 SX -683,40 U-135326 U-135326 -0-132931	T= 3 VOI.= .SY.SXY.S7= -627.00 -3.67842 -1.36148 1.36148	0.9144E-05 -137.17 S.1.=	-627.00 684.82	*0.17158E-04 -683.40 SIGE 630.41	2-D SOLID 4
46 NODES= 54 61 YC= 1.306 0 -3750E-02 G1.51G2.51G3= -137.17 ATIC FORCES ON NODE 61 ATIC FURCES UN NODE 61 ATIC FURCES UN NODE 53 ATIC FURCES ON NODE 53	00 53 MA TEMP= 453.9 SX -663.40 -0.135326 0.135326 0.135326 -0.135326	T± 3 VOL= .SY.SXY.S7= -827-00 -3.67842 -1.36148 1.36148	6.9144F-05 -137 517 5.1.=	-827.00 689.82	0.17155E-04 -683.40 SIGE= 630.41	ē-D SGLID 4
			152			

MC.YC= 1.306 0.2250 SIG1.SIG2.SIG3= -137	E-02 TEMP= 453,9	MAIE 5 VOL= SX,SY,SXY.S7= -827.00	0.9144E-05 -137.17 S-J.=	-827.00 689.82	0.36646E-04 -683.40	5-0 SOFID.
STATIC FUNCES ON NODE	55 -0.137931 67 0.135376 61 0.135376 54 -0.137931	-3.67842 -1.36148 1.36148 3.67842			0.36646E-04 -683.40 SIGE= 630.41	
## 48 NODES 56 #C.YC= 1.30b 57500 !SIG1.SIG2.SIG3 - 137 !STATIC FORCES ON NOCE	63 62 55 E=03 TEMP= 453,9 -17 =-683,40 56 =0-132931	MAT= 3 VOL= SX.SY.SXY.SZ= -827.00 -3.67842	0.9144E-05 -137.17 5.I.=	-827.00 689.82	0-19743E-04 -683.40 SIGE= 630.41	2-D SCLID
STATIC FUNCES ON RODE STATIC FUNCES ON RODE STATIC FORCES ON RODE	63 0.135326 62 0.135326 55 -0.132931	1.36148 1.36148 3.67842				The second second
STATIC FORCES ON NODE	E-02 TEMP= 444.4 2.7 1641.7 58 -0.135326 65 0.128051	SX.SY.SXY.S7= -134.26 -27810	0.1071E-04 -134.26 S.1.=	1641.7 1916.9	0.16884E-05 1782.7 STGE= 1850.5	5-D SOLID
STATIC FORCES ON NODE STATIC FORCES UN NODE TO NODES 59 CC.YE 1.311 0 6750	54 0.126051 57 -0.135326 66 65 58	-7.44396 -4.27810	0-1071E-04	The service distribution of the service space.		2-n snith
STATIC FURCES ON NODE STATIC FURCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	2.7 1641.7 59 =0.135326 66 0.128051 65 0.128051	4.27810 4.27810 7.44396 -7.44396	-134.26 S.I.=	1641.7 1916.9	0.24781E-05 1782.7 SIGE= 1850.5	2-0 30010
STATIC FORCES ON NODE TE ST NODES 60 C.YE 1.311 0.5250	58 -0.135326 67 66 59 E-02 TEMP= 44414	-4.27810 MATE 3 VDI.= SX.SY.SXY.S7=	0.1071E-04	1641.7	0 900195-06 1792 7	2-D SOLID
STATIC FORCES ON ADDE STATIC FORCES ON ADDE STATIC FORCES ON ADDE STATIC FORCES ON ADDE	60 -0.135326 67 0.128051 66 0.128051 59 -0.135326	-134-26 4-27810 7-44396 -7-44396	S.Ī.=	1916.9	0.90019E-06 1782.7 SIGE= 1850.5	A New York and A seekers - A Televis Control of the A Seekers and A Seek
11	58 67 60 E-02 TEMP= 444 μ 2.7 1641.7	MAT# 3 VOL= SX.SY.SXY.S7= -134.26	0.1071E=04 -134.26 S.I.=	1641.7	-0.90098E-06 1782.7 SIGE= 1850.5	S-D SOLID (
STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	61 -0.135326 68 0.128051 67 0.128051	4.27810 7.44396	The first and the second section of the	1710.7	1850.5	
STATIC FORCES ON NODE FI = 53 NODES= 62 62 WT.YC= 1.311 0.22508	60 -0-135326 59 68 61 5-02 TEMP= 444-4	-4.27810 MAT= 3 VOL= SX.SY.SXY.S7=	0.1071E-04 -134,26	1641-7	-0.24787E-05 1782.7	2=0 SULID (
STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	69 0.128051 68 0.128051 68 0.128051 61 -0.135326	4,27810 7,44396 -7,44396 -4,27810	Jetem	1710.7	-0.24787E-05 1782.7 STGE= 1850.5	
FI = 54 NODES = 03 WC.YC = 1.311 0.75006 SIG1.SIG2.SIG3 = 1786	70 69 62 -03 TEMP= 444 4 -7 1641-7	MAT= 3 VOL= SX.SY.SXY.S7=	0.1671E-04 -134.26 S.I.=	1641.7 1916.9	-0.16888E-05 1782.7 SIGE= 1850.5	2-0 SOLIO .
STATIC FURCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	70 0.128051 69 0.128051 62 -0.135326	7.44396 -7.44396 -4.27810	The state of the s	e distante distante distante distante distante distante del periodicio di successivo del conservo del conservo		
FI = 55 NODES = 65 %C.YC = 1.317 0.82506 SIG1.SIG2.SIG3 = 4604 SIG1IC FUNCES UN NODE	72 71 64 5-02 TEMP= 433.4 4-5 4478.3	MAT= 3 VOI.= SX.SY.SXY.S7= -119.23 16.2746	0.1229E-04 -119.23 S.I.=	4478.3 4723.7	0.20343E-00 4604.5 SIGE= 4661.9	2-0 SCLIO 7
STATIC FURCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	72 0-106570 71 0-106570 64 -0-128051	20.4291 -20.4291 -16.2746	in isaagganii			350 707 102 0-E
XC.YC= 1.317 0.6750E SIG1.SIG2.SIG3= 4604 STATIC FORCES UN NGDE STATIC FORCES UN NGDE	1.5 TEMP= 433 4 4.5 4478.3 	5X.SY.SXY.S7= -119.23 16.2746 20.4291	*119.23 \$.I.=	4478.3	0.70842E-06 4604.5 SIGE= 4661.9	2-0 30110 1
STATIC FUNCES UN NODE	65 -0.128051	-16.2746			0.45326E-06 4604.5 SIGE= 4661.9	
XC.XC= 1.317 0.52506 SIG1.51G2.51G3= SIATIC FORCES ON NODE STATIC FORCES ON NODE STATIC FORCES ON NODE	4478.3 67 -0.128051 74 0.106570 73 0.106570	16.2746 20.4291 -20.4291	\$.I.=	4723.7	SIGE= 4661.9	<u> </u>
SFATIC FORCES ON NODE ##	pe =0.158021	-1h_2/4h			-0.45282E-06 4604.5 SIGE= 46n1.9	
STATIC FUNCES ON NOCE STATIC FUNCES ON NOCE STATIC FUNCES ON NOCE STATIC FUNCES ON NOCE	75 0.106570 74 0.106570 67 -0.128051	16.2746 20.4291 -20.4291			•• •• • • • • • • • • • • • • • • • • •	
FI = 59 NCES 69 17 xc.yc= 1.317 0.22508 xf.yc=1.3163= 4604	75 58 -02 TEMP= 433.4 4.5 4478.3	MAT= 3 VOL= SX.SY.SXY.S7= -119-23	0.1229F-04 -119.23 \$.I.=	4478.3 4723.7	-0.70807E-06 4604.5 STGE= 4661.9	S-0.2CFID
STATIC FURCES UN NODE STATIC FORCES UN NODE STATIC FORCES UN NODE STATIC FURCES UN NODE	69 -0-128051 76 0-106570 75 0-106570 68 -0-128051	16.2746 20.4291 -20.4291 -16.2746			and the second of the second o	
Ft = 60 NGDES= 70 XC.YC= 1.317 0.7500E SIG1.SIG2.SIG3= 4604 STATIC FURCES ON NOCE	77 76 69 E-03 TEMP= 433.4 4.5 4478.3 70 -0.128051	MAT= 3 VOI.= SX.SY.3XY.S7= -119.23 16.2746	0.1229E-04 -119.23 S.I.=	4478.3 4723.7	-0.20'42CE-06 4604.5 SIGE= 4661.9	2-0 SCLID
STATIC FORCES ON NCCE STATIC FURCES UN NCCE STATIC FURCES ON NCCE	77 0.106570 76 0.106570 69 =0.128051	20.4291 -20.4291 -16.2746	A 17000100			
XC.YC= 1.324 0.82506 SIG1.SIG2.SIG3= 7778 STATIC FORCES ON NODE STATIC FORCES ON NODE	72 -0.106570 779 -0.657720F-0	SX.SY.SXY.SZ -87,409 32,9324 1 38,2181	-87.409 S.I.=	7678.0 7860.1	-0.12606E-06 7772.7 SIGE= 7813.2	5-0 SULTO (
STATIC FORCES ON NODE STATIC FORCES ON NODE	78 0.657720F-0 71 -0.106570	-34.2181 -32.9328 	0-1390E-04			2-0 SCLI0 1
MC VC = 1.324 STG1.SIG3-SIG3= 7772 STATIC FORCES ON ACCE STATIC FURCES ON ACCE STATIC FORCES ON ACCE	73 -0.106570 7678.0 77 -0.106570 79 0.657720F-01	32.937.6XY.57= -87_409 32.9324 1 36.2181 1 -38_2181	-8/.409 S.I.=	7678.0 7860.1	-0.35137E-06 7772.7 SIGE= 7813.2	a and the second
STAITC FUNCES ON NOVE FI = . AT NODES = 74 8	7. 92.1					

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TE 64 NUDESE 75 82 81 74 MA1= 3. VOI = 0.1390E-04
1C.YC= 1.324 0.3750E-02 IEMP= 421.0 SX.SY.SXY.SXT -87.409
SIG1.SIG2.SIG3= 77772.7 7678.0 -87.409 S.I.=
STATIC FUNCES UN NOCE 75 -0.106570 32.9328
STATIC FUNCES UN NOCE 62 0.657720F-01 36.2181
STATIC FUNCES ON NOCE 74 -0.106570 -32.9328
                                                                                                                                                                                                         2-D SOLID
                                                                                                                                                 0.20249E-06 7772.7
SIGE= 7813.2
                                                                                                                              7678.0
                                                                                                                              7860.1
FI = 65 NODES= 76 83 82 75 MAT= 3 VOL= 0.1390E-04 XC.YC= 1.324 0.2250E-02 TEMP= 421.0 SX.SY.SXY.S7= -87.409 SIGI.SIG2.SIG3= 7772.7 76 -0.106570 32.9328 STATIC FORCES ON NODE 83 0.657720F-01 38.2141 STATIC FORCES ON NODE 83 0.657720F-01 38.2141 STATIC FORCES ON NODE 75 -0.106570 -32.9328
                                                                                                                                                                                                          2-0 SOLID
                                                                                                                              7678.0
7860.1
                                                                                                                                                 0.35126E+06 7772.7
SIGE= 7613.2
FI = 66 NODES = 77 84 83 76 MAT = 3 VOL = 0.1390E-04 XC.YC = 1.324 0.7500E-03 TEMP= 421.0 SX.SY.SXY.SZ = -87.409 SIG1.SIG2.SIG3 = 7772.7 77.0106570 32.9328 STATIC FORCES ON NODE 84 U.557720F-01 38.2141 STATIC FORCES UN NODE 85 U.657720F-01 38.2141 STATIC FORCES UN NODE 85 U.657720F-01 38.2141 STATIC FORCES UN NODE 76 -0.106570 -32.9328
                                                                                                                                                 0.12589E-06 7772.7
SIGE= 7813.2
2-U SCLIU
                                                                                                                                                 0.25803E-07 113
S1GE= 11290.
                                                                                                                                                                           11277.
 ~ 2-6 SGL1D
FI =
                                                                                                                                                 0.73915E-07 1
SIGE= 11290.
                                                                                                                                                                           11277.
 TI = 69 NODES= 81 88 87 80 . MAT= 3 VOL= 0.1553F-04 (C.YC= 1.331 0.5250F-02 TEMP= 407.2 SX.SY.SXY.SXY.SXT -33.707 SIG1.SIG2.SIG3= 11277. 11278. -33.707 S.I.= STATIC FURCES ON NCDE 81 -0.657720F-01 54.8857 STATIC FURCES ON NCDE 88 0.902785F-12 61.4477 STATIC FURCES ON NCDE 87 -0.257720F-01 -54.8857
                                                                                                                                                                                                           2-0 SCLID
                                                                                                                                                 0.41783E-07 11277.
SIGE= 11290.
 F1 = 70 NODES=
                                                                                                                                                                                                           2-D SCL10
                                                                                                                                                      41765E+07 11277.
E= 11290.
                                                                                                                              11236.
FI 71 NODES 83 90 89 87 7 MAT 3 VOL 0.1553E-04
XC,YC= 1.331 0.250E-02 TEMP= 407.2, SX.SY.SXY.S7= -33.707
SIGL.SIG2.SIG3= 11277. 11236. -33.707 S.I.=
STATIC FORCES ON NODE 83 -0.657720F-01 54.8457
STATIC FORCES ON NODE 90 -0.284750E-09 61.4477
STATIC FORCES ON NODE 82 -0.657720F-01 -54.8857
                                                                                                                                                                                                           a-D SOLID
                                                                                                                                                             91E-07 11277.
11290.
FT = 72 NUDES = 84 91 90 A3 MAT = 3 VOL = 0.1553F-04
YC.YC = 1.331 G.7500E-03 TEMP= 407.2 SX.SY.SXY.S7 = -33.707
SIG1.SIG2.SIG3 = 11277. 111276. 27.707 S.I. =
STATIC FORCES ON NODE 84 -0.657720E-01 54.8857
STATIC FORCES ON NODE 91 0.784262E-12 61.4477
STATIC FURCES ON NODE 90 0.247654F-09 -61.4477
STATIC FURCES ON NODE 83 -0.657720E-01 -54.8857
                                                                                                                                                                                                           2-D SCLID
                 ***** REACTION FORCES ****
                                                                                IIME = 0.00000F+00 LOAD STEP= 1 ITERATION= 10 CUM. ITER.=
             NOTE - REACTION FORCES ARE IN THE NODAL COORDINATE SYSTEM
                  NODE
                   14
                                                                   65.1837
35.6729
20.1778
                   21
                  28
35
                   42
                   56
                  63
70
                                                                  -23.<u>7</u>185
          TOTAL 0.000000E+00 0.465761E-11
                                                 1 ITER 10 COMPLETED' TIME= 0.000000E+00 KUSE= 0 CUM. ITER.=
              *** LOAD STEP
          FNO OF INPUT ENCOUNTERED ON FILE?7
            ***** INPUT FILE SWITCHED FROM FILE27 TO FILE18
          PRINTOUT RESUMED BY JOOP
           **** ROUTINE COMPLETED **** CP = 69.440
           VEOR ENCOUNTERED ON FILE18
           **** RUN COMPLETEC **** CF= 69_4900 TIME=
                                                                                                                            10.1689
```



SWANSON ANALYSIS SYSTEMS. INC. HOUSTON, PENNSYLVANIA 15342 PHONE (412)746-3304 TWX 510-690-8655

THRUST CHAMBER MODEL

10.1721 5/ 8/84 CP= 4.770 ANSYS POSTRI NODAL POINT STRESSES IGAD STEP= 1 ITERATION= 10 TIME= 0.00000E+00 SXY · SZ NODE SX SIG1 SIGR SIGE -26.387 -549060.18282E-03 -549060.56382E-03 -54906-0.14688E-09 -54906-0.16282E-03 -54906-0.16282E-03 -54906-0.16282E-03 -54906-0.16282E-03 -54906-0.16282E-03 -54906-0.131478E-03 -380844-0.14082E-03 -380844-0.14082E-03 -380844-0.14082E-03 -380844-0.14082E-03 -380844-0.14082E-03 -380844-0.13478E-03 -380844-0.31478E-03 -380844-0.31478E-03 -380844-0.31478E-03 -380844-0.37473E-03 -22523-0.72110E-03 -22523-0.72110E-03 -22523-0.72110E-03 -22523-0.72110E-03 -22523-0.74078E-04 -23606-16.887 -23606-16.887 -23606-0.12617E-03 -28622-0.68308E-03 -24003-0.68308E-03 -24003-0.68308E-03 -26822-0.68308E-03 -26822-0.68308E-03 -26822-0.71777E-03 -28822-0.68308E-03 -26822-0.71777E-03 -28822-0.71777E-03 -2 1 -1-8897 2 -1-8852 3 -1-8880 4 -1-8876 5 -1-8895 6 -1-8852 7 -1-8897 -1.8770
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***** RUN COMPLETED ***** CP= 5.6800 TIMF= 10.1725

```
YAXZVMS ATONY ATONY VAXZAV SMVXAV
                                                                                                      8-MAY-1984 10:27
8-MAY-1984 10:27
8-MAY-1984 10:27
                                                                                                                                                USER3: [ANSYS.TUNYIO.LUG:1
USER3: [ANSYS.TUNYIG.LUG:1
USER3: [ANSYS.TUNYIG.LUG:1
                                                                                               000
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                                                    A A A A A A A A A A A A A
                                                    8-MAY-1984 10:10
8-MAY-1984 10:10
8-MAY-1984 10:10
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SCURVER = 1
SCURVE
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APPENDIX 3

PHOTOGRAPHS OF TEST SPECIMENS

This appendix is supplied to document the coating test specimens used for this program through photographs.

Figure 90 shows two flat and one ring specimen before testing.

Figure 91 shows two flat bend specimens after testing. The top specimen was bent with the coating in tensions; the bottom specimen with the coating in compression. The flake shown with the bottom specimen is a piece of coating which spalled from the middle span.

Figures 92 and 93 show close-ups of two bend specimens after test. Figure 92 shows a coating failure due to compressive stress and Figure 93 shows a coating which has been subjected to substantial tensile bending stress.

Figure 94 shows four bend specimens after test, and Figure 95 shows a close-up of one of these.

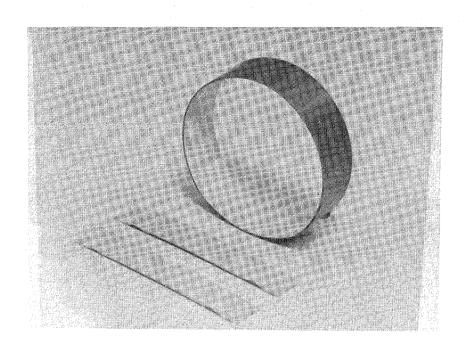


Figure 90. Specimens Prior to Testing

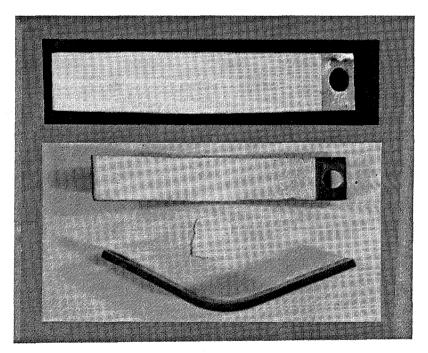


Figure 91. Two Bend Specimens Ater Test
Upper: Coating in Tension
Lower: Coating in Compression

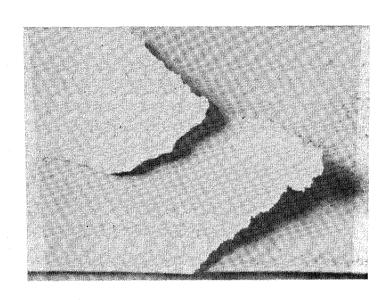


Figure 92. Compressive Coating Failure

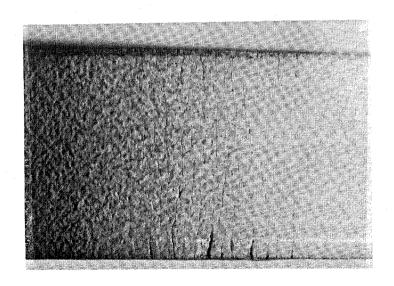


Figure 93. Tensile Coating Cracks

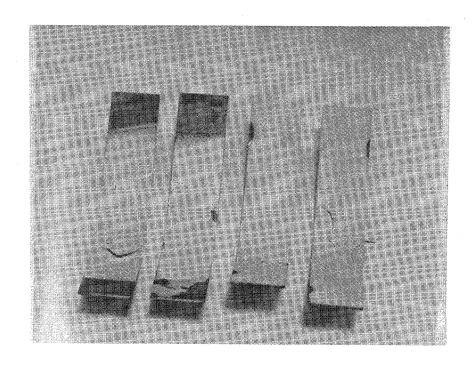


Figure 94. Specimens After Test

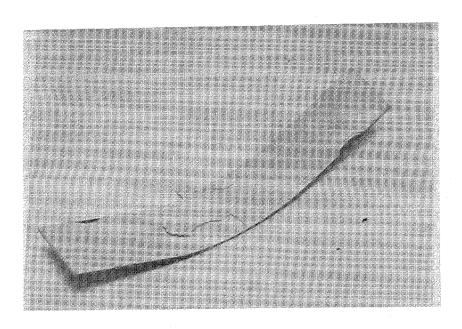


Figure 95. Bend Specimen After Test

