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# Computational Analysis and Preliminary Redesign of the Nozzle Contour of the Langley Hypersonic CF $_{4}$ Tunnel 

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## Summary

A computational analysis, modification, and preliminary redesign study was performed on the nozzle contour of the Langley Hypersonic $\mathrm{CF}_{4}$ Tunnel. Objectives of this work were, first, to study the origination and propagation of centerline disturbances measured at the exit and downstream of the exit in the existing nozzle; second, to determine the feasibility of modifying the present nozzle contour to eliminate the centerline disturbances; and, last, to redesign a nozzle contour for the $\mathrm{CF}_{4}$ Tunnel for different reservoir and exit conditions using the original design methodology.

This study showed that the present nozzle of the $\mathrm{CF}_{4}$ Tunnel was contoured incorrectly for the design reservoir and exit conditions and, in general, the diameter of the current nozzle was found to be too large over the length of the tunnel. The formation of disturbances, which are focused on the nozzle centerline, was predicted in terms of pitot pressure for the nominal and near-nominal reservoir conditions, and the predicted disturbance pattern across the nozzle exit was in agreement with measurements. A modified contour for the nozzle of the $\mathrm{CF}_{4}$ Tunnel was designed for the portion downstream of the maximum turning angle. This new contour theoretically provided a uniform exit flow for the nominal reservoir condition. Attempts to modify the current nozzle contour in the throat region to produce uniform exit flow for the nominal reservoir condition were unsuccessful. Discussion of that modification effort is included for reference. New nozzle contours were designed for the $\mathrm{CF}_{4}$ Tunnel to provide uniform flow at the nozzle exit with a Mach number and Reynolds number combination which closely matches that attainable in the Langley 20-Inch Mach 6 Tunnel. Two nozzle contours were designed: one having the same exit radius but a larger mass flow rate than that of the existing $\mathrm{CF}_{4}$ Tunnel, and the other having the same mass flow rate but a smaller exit radius than that of the existing $\mathrm{CF}_{4}$ Tunnel.

## Introduction

The subject of the present report originates from the calibration results of the Langley Hypersonic $\mathrm{CF}_{4}$ Tunnel reported by Midden and Miller (refs. 1 and 2). Of primary interest in this calibration are the centerline disturbances that were measured in the test section for most reservoir pressures and temperatures. Pitot pressure and total temperature surveys revealed that the flow at the nozzle exit was uniform only for reservoir temperatures greater than $1460^{\circ} \mathrm{R}$ with only a slight sensitivity to changes in stagnation pressure between roughly 1500 and

2500 psi. From those measurements, the best flow condition was found at the maximum stagnation temperature and pressure, which is $T_{t, 1} \geq 1460^{\circ} \mathrm{R}$ and $p_{t, 1}=2500$ psi, respectively. The appearance of centerline disturbances was generally observed at lower reservoir temperatures. However, lowering the reservoir temperature has become a necessity for the routine operation of the $\mathrm{CF}_{4}$ Tunnel since flow contamination became a serious problem when the tunnel was operated at high temperatures. This contamination posed a threat to the accuracy of test results, particularly heat-transfer studies, and to model surface instrumentation such as thin film gauges. To minimize these risks, the nominal operating temperature was lowered. As a result, the centerline disturbances are a constraint that must be considered when testing in the $\mathrm{CF}_{4}$ Tunnel. At present, the only way to avoid this problem is to position the model in the inviscid test flow between the centerline disturbance and the nozzle boundary layer. In this region, the test-section flow has been shown to be uniform (ref. 2) and free from serious disturbances; however, the diameter of the available test core in the $\mathrm{CF}_{4}$ Tunnel is now reduced by at least one-half. Consequently, the physical size of models tested in the tunnel is severely restricted and particular attention must be paid to the model orientation (angle of attack and yaw) within the reduced core of uniform flow.

In the present study, the problems and limitations of the current $\mathrm{CF}_{4}$ Tunnel have been addressed from the standpoint of hypersonic nozzle design. Three objectives were undertaken in this study and are presented in this report. First, an analysis of the existing $\mathrm{CF}_{4}$ Tunnel nozzle was performed to verify the original nozzle design and to determine the origination and propagation of the measured disturbances. The second objective was to determine if minor modifications to the existing nozzle contour would provide uniform flow at the nozzle exit for the design or for any new operating condition. Last, a preliminary design effort was undertaken to produce an entirely new nozzle contour for the $\mathrm{CF}_{4}$ Tunnel for new reservoir and exit conditions. For this last objective, the major design parameters such as maximum turning angle and prescribed centerline Mach number distribution were unchanged from those used to design the existing $\mathrm{CF}_{4}$ nozzle. The reasons for not modifying those parameters were twofold. First, the overall dimensions (i.e., nozzle length and exit diameter) were to remain similar to the existing nozzle. Second, rather than undertake a complete parametric study, the primary goal was to make a review exercise of the design procedure. As such, the nozzle designs presented in
this paper are preliminary and several items deserve further study.

| Symbols |  |
| :---: | :---: |
| $A, B$ | constants in wall temperature equation (see eq. (1)) |
| $C_{1}, C_{2}, C_{3}$ | virial coefficients |
| H | enthalpy, Btu/lbm |
| M | Mach number |
| $\dot{m}$ | one-dimensional mass flow rate through nozzle throat, $\mathrm{lbm} / \mathrm{sec}$ |
| $N_{\text {Re }}$ | Reynolds number, $\mathrm{ft}^{-1}$ |
| $p$ | pressure, psi |
| $R$ | gas constant for $\mathrm{CF}_{4}$, $565.2 \mathrm{ft}^{2} / \mathrm{sec}^{2} /{ }^{\circ} \mathrm{R}$ |
| $r$ | nozzle radius, in. |
| $T$ | temperature, ${ }^{\circ} \mathrm{R}$ |
| $T_{r}$ | gas temperature at nozzle throat, ${ }^{\circ} \mathrm{R}$ |
| $u$ | velocity, $\mathrm{ft} / \mathrm{sec}$ |
| $u_{\ell}$ | limiting velocity, $\sqrt{2 H_{t, 1}}, \mathrm{ft} / \mathrm{sec}$ |
| $x$ | axial distance, in. |
| $z$ | compressibility factor |
| $\gamma$ | ratio of specific heats |
| $\delta$ | boundary-layer thickness, in. |
| $\delta^{*}$ | boundary-layer displacement thickness, in. |
| $\rho$ | density, slug/ft ${ }^{3}$ |

Subscripts:

| $e$ | nozzle exit |
| :--- | :--- |
| ref | reference condition |
| $t, 1$ | reservoir stagnation condition |
| $t, 2$ | stagnation condition behind normal <br> shock |
| $w$ | nozzle wall |
| 2 | static condition at nozzle exit |

Superscript:

* condition at nozzle throat

Abbreviations:
BL boundary layer
\&
MOC method of characteristics

## Computational Methods

The present numerical designs were performed using the computer codes of references 3 and 4 that solve the inviscid, irrotational flow equations by the method of characteristics (MOC). The designs were corrected by using boundary-layer solutions along the nozzle wall to account for viscous displacement effects. MOC and coupled boundary-layer solutions were also used to analyze the flow field in the existing $\mathrm{CF}_{4}$ Tunnel nozzle. This approach is useful for the numerical analysis of internal flow problems but can be inadequate in some cases. The following sections describe the computational methods employed in this study in more detail.

## Design Method of Characteristics

The design method of characteristics program was originally developed by Johnson et al. (ref. 3) and was later modified (ref. 4) to improve the numerical procedure and to add some additional capabilities. It was shown (ref. 4) that the modified solution procedure eliminated a small inflection in the nozzle contour in the area of maximum turning that had been present in earlier solutions. It was also shown that the modifications had a minimal effect on nozzle shape (less than a 2 -percent change in nozzle radii downstream of maximum turning angle) so that the two codes produced nearly identical nozzle contours for the same set of design parameters. The inviscid contour of the existing $\mathrm{CF}_{4}$ nozzle was designed with the earlier MOC program (i.e., ref. 3); but in the present study, the latest version (ref. 4) was used.

Johnson's program (ref. 4) is applicable to the inviscid design of both axisymmetric and twodimensional supersonic nozzle contours and includes the capability of treating thermally and calorically imperfect gases in thermodynamic equilibrium. Design parameters available to the user include the reservoir conditions, the maximum turning angle of the nozzle, the centerline Mach number distribution in the initial expansion region, and the exit Mach number. For an imperfect-gas calculation, tabulated values of density ratio ( $\rho / \rho_{t, 1}$ ) and limiting velocity ratio ( $u / u_{\ell}$ ) as functions of Mach number are required for an isentropic expansion beginning at the design reservoir condition. In the design procedure, a Mach number distribution is first established along the nozzle centerline from the throat to
the desired exit Mach number. The final Mach line is constructed at this point, and then the program proceeds ("begins marching") backward toward the nozzle throat, solving the inviscid irrotational flow equations along left-running characteristics. The final inviscid contour is obtained by equating the mass flow through the nozzle along each computed characteristic.

A further discussion of the design procedure is warranted here since the design parameters chosen by the user can have a significant impact on the overall design. Figure 1(a) shows a diagram outlining the various regions used in the present design process for a supersonic nozzle. Region I is considered to be the throat region, region II is an area of radial or source flow, region III is a simple flow region where expansion waves that strike the wall are cancelled, and the last, region IV, is one of uniform flow. The line DE is the final Mach line with the Mach number at point D equal to the design exit Mach number. Point C is the inflection point in the nozzle contour and thus is the location of the maximum turning angle. In this design method, region II is approximated by source flow, and so the centerline Mach number distribution between points B and D can be determined by solving the radial flow equations. The Mach number distribution from A to B can then be prescribed between the value computed at point B and Mach 1 at point A.

Three options exist in Johnson's program (ref. 4) for describing the Mach number variation along line AB. First, a linear distribution can be used that results in a finite axial Mach number gradient at the throat (point A) and a relatively short throat section. The second option fits a third-order polynomial for the Mach number between A and B with a zero gradient constraint on Mach number at point A. A third available distribution uses a second-order polynomial for the Mach number variation and results in a finite, but controlled, axial gradient at the throat and also the shortest throat section of the three options.

Once the Mach number distribution is prescribed from A to B and computed from B to D, the only remaining initial data to be defined are along the final Mach line DE. These data are determined easily since the properties along DE are constant and are equal to the exit conditions. With this, the initial value problem is completely defined and may be solved in a marching manner. As described before, the program starts at line DE and proceeds backward toward the nozzle throat, computing each left-running characteristic from the centerline to the point where the stream function (i.e., the mass flow) is equal to the value at point $C$ that was computed using the radial flow approximation. Once found, the point on each
left-running characteristic then becomes a coordinate for the inviscid nozzle contour.

The two input parameters that have the most effect on the overall nozzle dimensions, for fixed reservoir and exit conditions, are the maximum turning angle and the centerline Mach number distribution in the initial expansion region (from points A to B ). Both parameters can vary the expansion region and, therefore, the overall nozzle length in a substantial manner. Decreasing the maximum turning angle has the effect of lengthening the expansion region and vice versa. It is generally accepted that a smaller turning angle provides a better chance of obtaining uniform flow because of a more gradual expansion. In the existing $\mathrm{CF}_{4}$ Tunnel, the nozzle was designed for a maximum turning angle of $16^{\circ}$, and this parameter was kept constant for the present calculations. With regard to the centerline Mach number distribution, the first of the options listed above (i.e., the linear variation) was used for the original $\mathrm{CF}_{4}$ nozzle design and is also used in this report. Keeping these two parameters constant served the intended purposes of this paper. By not varying these values, a new nozzle could be designed with overall dimensions similar to those of the existing nozzle and, in doing so, the application of the design process could be easily demonstrated. A more detailed design study should include a careful examination of these parameters.

It is probably true that the third-order polynomial provides a better distribution of centerline Mach number since this fit matches the gradients at points A and B and provides a necessary condition for a straight sonic line at the throat; however, this Mach number variation results in a lengthy throat section. In addition to physical length constraints, the authors in reference 4 discuss other possible problems associated with a long and narrow expansion section. They point out that the heat loads associated with such a design may exceed normal structural limits and, therefore, need to be considered. That study also examined the effect of test gas on nozzle design and showed that a low gamma gas (that is, one having a low ratio of specific heats) such as tetrafluoromethane $\left(\mathrm{CF}_{4}\right)$ inherently produces a long and narrow expansion region and thus compounds the problems.

## Analysis Method of Characteristics

In addition to design applications, the method of characteristics (MOC) can also be used for the analysis of supersonic internal flows for a given nozzle contour. For direct flow field calculations, the MOC requires different initial data from that for the design MOC and can involve difficulties associated with the formation of shock waves within the flow field.

In the present study, the basic analysis MOC code was taken from the text by Zucrow and Hoffman (ref. 5) and was modified for the $\mathrm{CF}_{4}$ gas and existing nozzle shape. In this code, an initial line is computed at the throat by starting with the closedform solution of Sauer (ref. 6) to obtain the sonic line. Sauer's method uses the transonic small perturbation equation and a power series solution to determine the equation of the sonic line in a converging-diverging throat. Since the sonic line is inappropriate to start the method of characteristics, an initial value line is computed where the radial perturbation velocity is zero. This initial data line extends from the wall at the nozzle minimum to a point on the centerline slightly downstream of the throat as shown in figure 1 (b).

For the downstream solution, the flow field within the range of influence of the initial value line is computed first, and then the method "begins marching" toward the nozzle exit. In the marching procedure, a wall point is computed first by either extending a left-running characteristic until it intersects the fixed wall or by selecting points along the wall a priori and computing the intersecting left-running characteristics in an indirect fashion. After a wall point is computed, the solution proceeds along the rightrunning characteristic emanating from the wall until the nozzle centerline is reached. The solution continues downstream in this manner, computing from the wall to the nozzle centerline until the right-running characteristics cross the exit plane where they are terminated.

A difficulty in using the method of characteristics for analysis of a given nozzle shape is the possible formation of oblique shock waves within the flow field. Formation of an oblique shock is indicated by the crossing of two or more characteristics of the same family, and when this occurs, several options are available. First, the characteristics can be allowed to overlap and form a multivalued solution in the flow field. Alternatively, a characteristic can be terminated when crossing of the same family occurs. Another possibility is to fit an oblique shock wave within the flow field using shock jump relations. The most preferable way to handle crossing characteristics would be the latter option; however, the present formulation is not suited for this procedure because of the irrotational equations employed. Therefore, in the present analysis, the characteristics are allowed to overlap or they are terminated if the crossing prevents a complete solution.

## Viscous Displacement Effects

The inviscid flow field computed by either method of characteristics described above must be coupled
with viscous contributions near the nozzle wall to be accurate for design or analysis purposes. This requirement is true for the present hypersonic nozzle of interest since the boundary-layer growth is large over the relatively long nozzle. In earlier work, the integral boundary-layer method of reference 3 was modified by Hunt and Boney (appendix B of ref. 7) to account for the real-gas behavior of $\mathrm{CF}_{4}$ gas due to vibration and high-pressure effects, and this method was employed to compute the viscous displacement effects. This code was coupled with the MOC program of reference 3 as part of the design procedure. In the present study, the integral boundary-layer code of reference 7 was only used to check the previous nozzle design. In the analysis of the existing nozzle and for the present design work, viscous displacement effects were computed using a finite-difference boundary-layer code developed by H. Harris Hamilton II of the Langley Research Center.

Boundary-layer edge conditions for the viscous calculations ( $x, r, M, u, p$, and $T$ ) are obtained directly from the inviscid solution at the wall. At the solid surface, a steady-state wall temperature is computed with the analytic equation given by

$$
\begin{equation*}
T_{w}=\frac{T_{r}-A\left(r / r^{*}\right)^{1.8}}{1+B\left(r / r^{*}\right)^{1.8}} \tag{1}
\end{equation*}
$$

where $T_{r}$ is the temperature of the gas at the nozzle throat. The constants $A$ and $B$ can be determined by defining the wall temperature at a point where $r / r^{*}$ is large compared to unity and also at the nozzle exit. Equation (1) was presented in reference 3 in conjunction with the design of a hypersonic nitrogen tunnel. As noted in that study, this equation is expected to give a reasonable variation of wall temperature. For all calculations in this study, the boundary-layer and displacement thicknesses are assumed to be zero at the nozzle throat, and the viscous layer is assumed to be turbulent for the length of the nozzle. Turbulence is modeled in the finite-difference code by a two-layer eddy viscosity model (ref. 8) with instantaneous transition at the nozzle throat.

Implementing the viscous displacement effect on the design of a nozzle contour is usually done in a straightforward manner. In this study, a single pass with the boundary-layer code was made after the inviscid contour was computed and the conditions at the inviscid wall had been saved for input as the boundary-layer edge conditions. The computed displacement thicknesses were then added to the inviscid contour to generate the final nozzle coordinates. No smoothing of the inviscid or viscous contours was performed.

To correct the inviscid solution for the case where an analysis of a given nozzle shape is performed requires an iterative procedure. In this case, the process begins with an inviscid solution for the physical nozzle contour which, in actuality, already accounts for viscous displacement. In a manner similar to the design procedure, a boundary-layer solution is obtained along the nozzle wall using the inviscid solution to provide the edge conditions. The displacement thicknesses from the viscous solution are then subtracted from the original nozzle contour to provide a new "apparent wall." A second inviscid solution uses the apparent wall as the solid boundary and thus provides improved boundary-layer edge conditions. Next, a second boundary-layer solution is calculated along the original nozzle wall but using the new edge conditions. The resulting displacement thicknesses are again subtracted from the original nozzle contour to provide a better apparent wall. The iteration between inviscid and boundary-layer solutions as described above is continued until the displacement thicknesses (i.e., the apparent wall) are converged to within a plottable difference of approximately 3 percent. This process was generally found to require only two iterations for convergence in the present cases.

## Thermodynamic and Transport Properties of $\mathrm{CF}_{4}$ Gas

At high pressures, the $\mathrm{CF}_{4}$ gas behaves in an imperfect manner, and relations for the thermodynamic and transport properties must account for this behavior. For present reservoir conditions, $\mathrm{CF}_{4}$ remains undissociated as it is heated and expanded through the nozzle, but intermolecular force effects are significant in the reservoir and nozzle throat region where the pressures and densities are large. It is assumed in this study that the $\mathrm{CF}_{4}$ gas remains in vibrational equilibrium (see appendix A in ref. 7) and that the flow in the inviscid region of the nozzle is isentropic. Consequently, the thermodynamic properties can be tabulated for values of Mach number by performing a one-dimensional expansion from the reservoir conditions to specified values of temperature. The computer code used to perform the isentropic expansion is based on the expressions and thermodynamic equations for $\mathrm{CF}_{4}$ given by Hunt and Boney (ref. 9). This same code is currently used to determine test-section flow conditions in the $\mathrm{CF}_{4}$ Tunnel (ref. 2). Its use is beneficial in this study since comparisons between calculated flow conditions and the present analysis will be on a consistent basis.

The finite-difference boundary-layer code of Hamilton uses the analytic expressions developed by Sutton (ref. 10) for the thermodynamic and trans-
port properties of $\mathrm{CF}_{4}$. These equations are simpler to apply and require less computer time than the full equations of reference 9 . They are accurate to within $\pm 5$ percent for thermal conductivity and specific heat and to less than $\pm 1$ percent for the other properties.

The thermally perfect equation of state used in reference 10 was modified for this study to account for high-pressure effects encountered in the throat region. The equation of state developed for this study has the form

$$
\begin{equation*}
p=z \rho R T \tag{2}
\end{equation*}
$$

where

$$
z=1.0+C_{1} \bar{\rho}+C_{2} \bar{\rho}^{2}+C_{3} \bar{\rho}^{3}
$$

with

$$
\begin{aligned}
C_{1}= & 0.5238-428.4 / T-37920.0 / T^{2} \\
& -1.720 \times 10^{7} / T^{3} \quad\left(\mathrm{ft}^{3} / \text { slug }\right) \\
C_{2}= & 0.02465+152.9 / T-1.044 \times 10^{5} / T^{2} \\
& +4.034 \times 10^{7} / T^{3} \quad\left(\mathrm{ft}^{6} / \text { slug }^{2}\right) \\
C_{3}= & 7.086 \times 10^{-5} T-0.03474 \quad\left(\mathrm{ft}^{9} / \mathrm{slug}^{3}\right)
\end{aligned}
$$

and

$$
\bar{\rho}=p / R T
$$

The full expressions given in reference 9 were also incorporated into the finite-difference boundarylayer program to check the validity of using the approximate relations. Comparisons of boundary-layer solutions using both methods to obtain thermodynamic and transport properties showed differences of less than $\pm 3$ percent in displacement thicknesses. As a result, the approximate relations were considered adequate for use in the present boundary-layer calculations.

## Results and Discussion

The objectives of the current study, as stated in the "Introduction," were as follows: (1) To analyze the existing $\mathrm{CF}_{4}$ nozzle, (2) to determine if the existing nozzle could be modified to provide uniform test-section flow for nominal reservoir conditions, and (3) to redesign a new nozzle contour for different reservoir and exit conditions. Results for each objective are discussed in detail in the following subsections.

## Analysis of Existing CF 4 Nozzle

Radially uniform test-section flow is obtained in the $\mathrm{CF}_{4}$ Tunnel only when the reservoir temperature is equal to or greater than $1460^{\circ} \mathrm{R}$. As discussed in reference 2, the contamination level in the flow increases with increasing temperature. Therefore, to
protect the tunnel and models and to provide a wider Reynolds number range, it is desirable to operate at lower stagnation temperatures. However, at lower $T_{t, 1}$, disturbances are created in the internal flow field and appear on the nozzle centerline. Midden and Miller (refs. 1 and 2) have measured the disturbances at the nozzle exit and downstream of the exit. In some cases, the disturbance is characterized by a "spike" in the total pressure profiles measured across the nozzle exit, whereas other slightly different conditions have made the disturbance appear as a "dip" in total pressure at the centerline. The dip or spike character at the nozzle exit depends on the reservoir conditions, but it can alternate with axial distance downstream of the nozzle exit for a fixed reservoir condition. The local dip or spike in total pressure at the nozzle centerline has been measured to be generally 15 percent lower or higher, respectively, than the average pressure across the test section. This variation has been shown (ref. 2) to produce erroneous measurements of heating rates, surface pressures, and bow-shock shapes.

The existing $\mathrm{CF}_{4}$ Tunnel nozzle was designed using the MOC program developed by Johnson et al. (ref. 3) and the modified boundary-layer code presented in reference 7. Since the revised version of this MOC code (ref. 4) was the primary design tool used in the present study, it was desirable to reproduce the existing nozzle contour as a check on the code and the existing nozzle. Calculations showed that from using the MOC code of reference 4 and the integral boundary-layer program described in reference 7 , the existing contour could be reproduced for a reservoir condition of 2500 psi and $1260^{\circ} \mathrm{R}$, an exit Mach number of 6 , and a maximum turning angle of $16^{\circ}$.

There has been some question in the past about the design reservoir condition for the existing $\mathrm{CF}_{4}$ Tunnel since it was not documented. It was generally assumed that the design condition was at 2500 psi and $1460^{\circ} \mathrm{R}$ since this was where the flow was most uniform. However, based on the present calculations, it appears that the actual design was performed for $p_{t, 1}=2500 \mathrm{psi}$, but with $T_{t, 1}=1260^{\circ} \mathrm{R}$. The question arises as to why the measured disturbances occur at the lower stagnation temperature but disappear at the higher, presumably off-design, reservoir temperature. To answer this, a detailed review of the design method of characteristics (ref. 4) and the integral boundary-layer code, as modified in reference 7 , was performed.

This review uncovered a subtle but significant error that was made in mating the inviscid and boundary-layer codes in the design process. The error arose from a mismatch of the reference
enthalpies used for thermodynamic properties. In the integral boundary-layer code, the equation used for static enthalpy across the layer was based on a reference enthalpy of $H_{\text {ref }}=0$ at $T_{\text {ref }}=0^{\circ} \mathrm{R}$. However, the total enthalpy at the boundary-layer edge, which was input, was based on the equations given in reference 9 that have a reference enthalpy of $H_{\text {ref }}=200 \mathrm{Btu} / \mathrm{lbm}$ at a reference temperature $T_{\text {ref }}$ of $820^{\circ} \mathrm{R}$. The inconsistent use of enthalpy between the layer and edge directly affected the calculation of density in the boundary layer and thus affected the integration for displacement thickness in the integral approach.

Correction of the enthalpy mismatch was made to the integral boundary-layer code in the present study, and subsequent calculations showed the predicted boundary-layer and displacement thicknesses at the nozzle exit to be as much as 40 percent less than had previously been computed. Thus, contours designed for the $\mathrm{CF}_{4}$ gas with the MOC and integral boundary-layer codes before this correction were generally too large in diameter, and this mismatch is apparently the major problem in the existing nozzle. Figure 2 shows a comparison of the existing nozzle contour with a new contour designed with the same MOC code but with the finite-difference boundary-layer program for the same design conditions ( $p_{t, 1}=2500 \mathrm{psi}$ and $T_{t, 1}=1260^{\circ} \mathrm{R}$ with $M_{2}=6$ exit flow). For this new contour, the enthalpy was based on a consistent reference in the inviscid and viscous programs. As shown, the existing nozzle radius is less than that predicted in the initial expansion region and eventually becomes larger than that predicted so that there is a large discrepancy in radius at the exit. The existing $\mathrm{CF}_{4}$ nozzle is slightly longer than the newly designed contour as shown in figure 2. This difference is due to a cylindrical section, approximately 11 in . long, that was added to the original nozzle design.

A simple illustration of the problem presently existing in the $\mathrm{CF}_{4}$ Tunnel can be made by considering the one-dimensional compressible flow theory assumed applicable for the inviscid flow at the nozzle exit. Figure 3 shows the nozzle-exit radius nondimensionalized by the throat radius as a function of Mach number according to the one-dimensional theory for various stagnation pressures and temperatures. These curves were obtained from the thermodynamic properties computed from an isentropic expansion using the equations of Hunt and Boney (ref. 9) and show the extreme sensitivity of Mach number to $r / r^{*}$ for the $\mathrm{CF}_{4}$ gas. For a design Mach number of 6 with $p_{t, 1}=2500 \mathrm{psi}$ and $T_{t, 1}=1260^{\circ} \mathrm{R}$, the inviscid radius ratio would be 38.25 according to this figure. This value can be compared with the
ratio for the existing nozzle, which is 44.85 as shown in the figure. This difference between inviscid radius and the existing radius would have to be made up by the boundary layer and, based on measurements (refs. 1 and 2), a boundary-layer thickness of this size at the nozzle exit is unrealistically large for the $\mathrm{CF}_{4}$ nozzle. Figure 3 also shows that at $M_{2}=6$ the reservoir conditions of $T_{t, 1}=1460^{\circ} \mathrm{R}$ and $p_{t, 1}=2500 \mathrm{psi}$ could not be correct for the existing $\mathrm{CF}_{4}$ Tunnel since there is no radial distance left to account for viscous displacement.

Some understanding of the centerline disturbances that exist in the $\mathrm{CF}_{4}$ Tunnel was gained by finding that the existing nozzle was not contoured correctly for the design reservoir and exit conditions. Those calculations, however, did not establish the origination of the measured disturbances, nor did they provide any insight into the possible contour modifications that may be made. For these reasons, the analysis MOC code was used to obtain flow field solutions in the nozzle for several reservoir conditions. The conditions chosen correspond to instances where the nozzle-exit flow was uniform and also to the conditions where a spike and a dip occurred in the total pressure profile. Reservoir conditions for these three cases were as follows, respectively: $p_{t, 1}=2552 \mathrm{psi}$ and $T_{t, 1}=1481.4^{\circ} \mathrm{R} ; p_{t, 1}=1742 \mathrm{psi}$ and $T_{t, 1}=1274.3^{\circ} \mathrm{R}$; and $p_{t, 1}=1515 \mathrm{psi}$ and $T_{t, 1}=1218.9^{\circ} \mathrm{R}$.

Wall temperature distributions along the nozzle were computed with equation (1) for use in the viscous part of the solutions. Since the actual $\mathrm{CF}_{4}$ nozzle is heated by resistance heaters in the throat section to approximately the reservoir temperature before each run, the input wall temperature distributions were made to fit this variation. This process was accomplished by setting $T_{r}$ equal to the reservoir temperature in equation (1). The wall temperatures at two axial stations were estimated from unpublished measurements made on the exterior nozzle wall during several tunnel runs by Raymond E. Midden of the Langley Research Center for use in the equation. Specifically, the wall temperature at the axial station where $r / r^{*}=20$ was set to $660^{\circ} \mathrm{R}$ for reservoir temperatures around $1260^{\circ} \mathrm{R}$ and was set to $760^{\circ} \mathrm{R}$ for reservoir temperatures closer to $1460^{\circ} \mathrm{R}$. The wall temperature at the nozzle exit was set to $530^{\circ} \mathrm{R}$ in all cases. Figure 4 shows the three input wall temperature distributions for the present cases. Note that each distribution begins at the respective reservoir temperature, remains constant for about 6 in., decreases rapidly to the value defined at the first station, and then approaches the temperature defined at the end of the nozzle. Also shown in this figure are the measured, exterior, nozzle wall temperatures for
tunnel conditions similar to the present, low reservoir temperature cases. The assumed variation of wall temperature is in agreement with the general trend of the experimental measurements.

Numerical calculations for the three cases were performed with the analysis MOC and finitedifference boundary-layer codes described previously. Two complete iterations were performed between the MOC and boundary-layer solutions for each case. Results from these calculations are shown in figures 5 and 6 . In figure $5(\mathrm{a})$, a representative computational mesh from a MOC solution is illustrated for one case, whereas figure $5(\mathrm{~b})$ shows the corresponding contour plot of Mach number. Figure 6 compares the predicted and measured total pressure ratio profiles across the test section at the nozzle exit for each case.

The computational mesh in figure 5(a) shows the actual left- and right-running characteristics computed as part of the MOC solution for the case where a spike in total pressure was measured at the nozzle exit. As can be seen, the mesh used was fairly coarse, especially in the downstream portion of the nozzle. The indirect method of computing the intersection of left-running characteristics with the predefined wall was used for these calculations so that some control over spacing could be achieved. Arbitrary refinement of the grid was not possible, however, because of the crossing of characteristics predicted in the flow field near the nozzle exit. By using a coarser grid, the number of crossing characteristics was limited and complete solutions were obtainable. Crossing of right-running characteristics generally occurred near the nozzle centerline and the nozzle exit for the three cases considered and indicated the formation of oblique shock waves within the nozzle. Note in figure 5(a) that the characteristic mesh is well-behaved in the throat and initial expansion region until $x=35 \mathrm{in}$. After the $35-\mathrm{in}$. station, the right-running characteristics alternately compress and expand for the remainder of the nozzle length.

Figure 5(b) shows the Mach contour for the same case where a spike in total pressure exists and clearly illustrates the disturbances in the flow field. In this and in the other cases, disturbances were found to emanate all along the nozzle wall downstream of $x=35 \mathrm{in}$. For this case in particular, there appears to be strong compressions or shock waves that strike the centerline between 65 in . and the nozzle exit ( $x=93$ in.). By tracing these disturbances backward with the aid of figure $5(\mathrm{a})$, it is observed that these waves originate from the nozzle wall between 35 and 45 in . The occurrence of these waves was noted for each case; however, the exact location and
the strength of the waves varied with the reservoir condition. In the first case, where the most uniform exit flow was measured, a weak compression wave was found to intersect the centerline at around 65 in . where it was reffected back toward the wall. For the second case (shown in fig. 5), where a spike in total pressure occurred, two strong waves are seen coming from the nozzle wall and intersecting the centerline. The first is reflected at 65 in . and the second intersects the centerline almost exactly at the nozzle exit. Note in figure 5(b) that the reflection of the first wave from the centerline intersects with a second wave coming from the wall to produce a rather complex flow. In the last case, where a dip in total pressure was measured, two strong waves were again found to emanate from the wall and intersect the centerline. This time, however, the second wave hit the centerline about 10 in . upstream of the nozzle exit.

The location of the compression or shock waves can be correlated with the measured centerline disturbances. In figure 6, the ratio of total pressures at the nozzle exit to the reservoir pressure is plotted across the test section from the centerline to the nozzle wall. Both measured and predicted values are included for comparison. In addition, experimental and predicted free-stream values of Mach number, static pressure, and temperature are compared. Note also that the predicted displacement and boundarylayer thicknesses at the nozzle exit are marked at the appropriate radius on the figure.

In figure 6(a), the total pressure profile is shown for the condition where the most uniform flow was measured. The experimental data in this case show a slight saddlelike distribution and a small rise in pressure on the centerline, but the data are fairly uniform across the test section. The inviscid pressure predictions from the analysis MOC code are in good agreement with experiment for a radius between 3 and 8 in . Closer to the centerline ( $r<3 \mathrm{in}$.), the agreement between prediction and experiment is not as good, and the computed solution shows a slightly larger rise in pressure on the centerline.

In the second case, a spike in pressure was measured on the nozzle centerline as shown in figure 6(b). Prediction and experiment are again in good agreement between 3 and 8 in . from the centerline, whereas the computation tends to overpredict the pressure closer to the centerline. However, the spike behavior of the measured data on the centerline is also exhibited in the calculation. Experimental data for the last case were obtained from Midden and showed similar results (fig. 6(c)). Namely, experiment and theory agree quite well across the test section except within 3 in . from the centerline. The general behavior of
the centerline disturbance is captured, however, and a dip in total pressure is predicted.

The poorer quantitative agreement between experiment and prediction in the innermost core of the test section ( $r<3 \mathrm{in}$.) for these cases is not too surprising since indications are that shock waves are formed and reflected in this region and a complex flow field evolves. In these areas, the present method of analysis cannot be expected to be accurate; however, the solutions do show good qualitative agreement. In correlating the flow field predictions with the measured disturbances, it appears that the spike in total pressure is caused by a strong compression (or shock) wave that intersects the centerline just upstream of the nozzle exit. For the case of a dip in pressure, the flow field solution indicates that a compression wave intersects the centerline well upstream of the exit and is followed by an expansion region that crosses the exit plane, thereby causing the local minimum in total pressure. The alternating dip and spike character of the centerline flow downstream of the nozzle exit measured by Midden and Miller (refs. 1 and 2) could also be explained in this context. In the case of the most uniform exit flow, it appears that the compression waves formed from the turning contour are not as severe, and the strongest waves do not intersect the centerline in the proximity of the exit plane.

The detailed flow field analysis performed for the three representative cases revealed the origination of the measured disturbances in the $\mathrm{CF}_{4}$ Tunnel. Most importantly, the analysis showed that the disturbances are not created in the upstream throat section of the nozzle but are instead formed from the nozzle wall downstream of the $x=35 \mathrm{in}$. station. All indications are that the flow expansion is too great initially because of the larger than necessary diameter of the contour for the near-nominal reservoir conditions. After excessive expansion, the flow is then turned nonisentropically through the remaining portion of the nozzle. At the higher reservoir temperature, the flow is not expanded too much judging by the maximum Mach number attained. Still, in this case the flow is not turned without creating some weak disturbances.

## Modification of Existing Nozzle

A major objective of this study was to design a modification to the existing $\mathrm{CF}_{4}$ Tunnel nozzle that would produce a uniform exit flow for reservoir conditions that were tolerable with regard to the contamination problem. If a simple contour modification could be devised, then the tunnel could realize its full capability, in terms of usable test-section flow, with minimum alteration. Since the existing
nozzle consists of four separate sections, it would be fairly easy to change the contour within a region by simply replacing the affected section. It was hoped further that a simple redesign of the throat section alone could be devised to produce the uniform flow since this is probably the least expensive modification. Unfortunately, these design objectives have not been fully met in this study; however, a discussion of the results of this effort may be useful and is included here.

Because of the nonisentropic turning of the flow in the current $\mathrm{CF}_{4}$ nozzle for the nominal reservoir conditions, it is not clear whether any modification can be made to the throat-section contour that would also satisfy the downstream contour. An easier alternative, but less preferable from a cost standpoint, would involve keeping the throat section fixed and recontouring the nozzle downstream of the maximum turning location. This modification has been accomplished numerically in the present study using the following reservoir and exit conditions: $p_{t, 1}=1500 \mathrm{psi}$, $T_{t, 1}=1260^{\circ} \mathrm{R}$, and $M_{2}=6$ exit flow.

The new contour was obtained by first performing a MOC calculation and coupled boundary-layer solution for the initial expansion region of the present nozzle that is to be kept fixed. Afterward, the remaining nozzle contour was designed using the initial data provided from the MOC solution. The results of this modification are shown in figure 7 where the existing $\mathrm{CF}_{4}$ nozzle contour is compared with the new contour design. The new nozzle contour is identical to the existing one up to the maximum turning angle. Downstream of that point, the new contour has a smaller radius than the current nozzle and the differences increase over the remaining length. This modification is in accord with the earlier findings where the existing nozzle contour was established to be generally too large in diameter. Note in figure 7 that the newly contoured section comprises 90 percent of the overall length of the nozzle.

A promising method for modifying supersonic nozzle contours has recently been developed by Erlebacher et al. (ref. 11) at the Langley Research Center for use in modifying the Langley 8-Foot HighTemperature Tunnel. The basic assumption in their approach is that the flow through a correctly designed inviscid contour can be reversed since it is an isentropic process. With this approach, the modification procedure begins at the nozzle exit where an inviscid solution is obtained for the reversed flow going into the nozzle toward the throat. Initial conditions for this solution are provided by specifying a uniform flow at the nozzle exit (now the entrance). Boundary conditions are set by the physical nozzle wall. The backward flowing inviscid solution is obtained up to
the last point on the nozzle wall that is to be unmodified. This inviscid solution then provides initial data to start a method of characteristics solution from the end of the unmodified nozzle wall toward the throat. Boundary data for that region are set by defining the remaining centerline Mach number distribution between the inviscid solution and the throat. With this definition and the initial data from the backward-flow solution, the method of characteristics can provide a new contour for the nozzle throat section. Viscous corrections are determined by sweeping forward in a normal manner with the new throat section and then repeating the backward-design process in an iterative manner.

In their work, Erlebacher et al. (ref. 11) used a full solution of the Euler equations for the backwardmoving flow field and for the forward-sweeping calculations. In the present study, these same ideas have been applied in an attempt to modify the throat section of the $\mathrm{CF}_{4}$ nozzle; however, the method of characteristics was employed to compute the backwardflow solution since the thermodynamics were already developed for the MOC code.

Several different stagnation and exit conditions were considered for the $\mathrm{CF}_{4}$ nozzle, and inviscid solutions for the backward flow were obtained up to the location of maximum turning. At this location, however, problems arose that Erlebacher et al. (ref. 11) did not encounter. Examination of the backwardflow solution revealed the existence of crossing characteristics emanating from the unmodified wall section. In reference 11, the authors discussed the possible formation of shock waves in the backward-moving flow field. Their conclusion is that the design process can continue with the complete Euler solution, and if the waves are weak enough, they may disappear when the viscous corrections are added. In the present case, the shock waves that were formed in the backward-flow field by the $\mathrm{CF}_{4}$ nozzle contour invalidate the MOC for use in the modification procedure. Continuing the modification process outlined in reference 11 will apparently demand using the full solution to the Euler equations. This process, which would require modification of the computer codes described in reference 11 to incorporate the imperfectgas behavior of $\mathrm{CF}_{4}$, has not been done in the present study.

Another approach to the modification of the $\mathrm{CF}_{4}$ Tunnel that has not been tried, but may be useful, is to study the effects of arbitrarily changing the throat or downstream section contours. One could intuitively devise new contours and then perform flow field analyses like those done in this paper to examine the effects on the exit flow. With this approach, it would likely be very difficult to
arrive at the modifications necessary to produce a uniform flow, but such a parametric study might provide insight.

## Design of New $\mathrm{CF}_{4}$ Nozzle

The last objective in this study was to design a nozzle contour for the $\mathrm{CF}_{4}$ Tunnel for reservoir and exit conditions different from those for which the nozzle was originally designed. Unlike the modification effort, this process entailed fewer constraints and was successful. The primary purpose of a new nozzle design is to match the conditions in the $\mathrm{CF}_{4}$ Tunnel more closely to the free-stream conditions of other test facilities. Matching the free-stream Reynolds numbers and Mach numbers of the $\mathrm{CF}_{4}$ Tunnel with those of another wind tunnel using air as the test gas is important since the effect of shock density ratio or gamma $(\gamma)$ may be studied independent of Mach number or Reynolds number differences. At present, the Langley 20-Inch Mach 6 Tunnel is the most appropriate facility for this purpose, and thus the new design of the $\mathrm{CF}_{4}$ nozzle was constrained to match the Reynolds number and Mach number of this facility. At the lowest reservoir pressure condition in the 20-Inch Mach 6 Tunnel, a test-section Mach number of 5.77 and a Reynolds number of $0.64 \times 10^{6}$ per foot are obtained (ref. 12). An isentropic expansion of the $\mathrm{CF}_{4}$ gas from reservoir conditions of $p_{t, 1}=1600 \mathrm{psi}$ and $T_{t, 1}=1260^{\circ} \mathrm{R}$ gives approximately the same Reynolds number for $M_{2}=5.77$. Therefore, for the present nozzle design, these values of reservoir and exit conditions were selected.

The nozzle design in this study was performed using the same inviscid technique used in the previous $\mathrm{CF}_{4}$ Tunnel design. The maximum turning angle was kept fixed at $16^{\circ}$, and the centerline Mach number variation in the throat section was defined using the linear fit described previously so that the overall nozzle dimensions would remain similar. For the viscous corrections, the finite-difference boundary-layer program of Hamilton was used. The boundary layer was assumed turbulent for the length of the new nozzle contour, and the wall temperature variation was defined using equation (1) in the same manner discussed previously. In effect, all design variables except reservoir and exit conditions were the same as those used to design the existing nozzle. Proceeding in this manner allowed direct comparisons of new designs with the existing nozzle and was sufficient for the purposes of this study.

An additional constraint on the nozzle design for the $\mathrm{CF}_{4}$ Tunnel involves the mass flow rate. Primarily for economic reasons (ref. 2), it is preferable to keep the same mass flow rate of the present tunnel in the new nozzle design. This requirement causes
a slight dilemma as shown in figure 8. In this figure the inviscid core radius of the $\mathrm{CF}_{4}$ nozzle exit is plotted as a function of the one-dimensional mass flow rate through the throat for the new reservoir and exit Mach number conditions. This figure shows that retaining the same mass flow rate as that of the existing tunnel operating at nominal reservoir conditions results in an inviscid core exit radius of about 6.4 in . This radius is less than the approximate 8 -in. inviscid radius currently provided in the $\mathrm{CF}_{4}$ Tunnel (assuming that there was no centerline disturbance). When the disturbance is taken into account by testing off the centerline, the existing inviscid radius then drops to about 4 in. and, thus, the present inviscid design offers some improvement. At the other extreme, retaining approximately the same inviscid radius as that of the current nozzle as designed (i.e., 8 in . by neglecting the centerline disturbance) requires a mass flow rate which is nearly 60 percent greater than that currently used. In the present study, new $\mathrm{CF}_{4}$ nozzles have been designed for both conditions: the first has about the same mass flow rate as that of the current tunnel, and the second has approximately the same inviscid exit radius as that of the existing nozzle.

Figure 9 shows the nozzle-wall temperature distributions used for the present designs, and the new nozzle contours computed using the MOC and boundary-layer codes are shown in figure 10. A magnified view of the throat region is presented in figure 11. A listing of the wall coordinates for both designs is presented in table I. These tables were obtained by cubic spline interpolation of the computed coordinates so that even axial spacing of the tabulated values could be presented. The radial distance at the throat $(x=0)$ was obtained by extrapolating the predicted contour backwards since a MOC solution was not obtained at that point. This result occurred because the sonic point on the nozzle centerline is slightly downstream of the nozzle minimum. The reservoir and predicted exit properties for the new $\mathrm{CF}_{4}$ nozzles are listed and compared with those from the 20 -Inch Mach 6 Tunnel in the table on page 11.

Comparing the new nozzle contours with the existing $\mathrm{CF}_{4}$ nozzle, which has a design error, showed substantial differences in both radius and length for the case where the mass flow rate is approximately equal. In that case, the throat radius is identical to the existing nozzle ( $r^{*}=0.2229 \mathrm{in}$.), but the total length is reduced from the present 93 in. to 70.4 in., and the physical exit radius is 7.89 in . as compared with 10 in . The throat radius of the contour, which results in approximately the same inviscid exit radius, is only slightly greater ( $r^{*}=0.2792 \mathrm{in}$.). The
overall length in that case was about 88 in ., and the physical exit radius was computed to be 9.85 in . Note that the larger throat radius results in a longer and narrower initial expansion region as shown in figure 11.

| Property | $\mathrm{CF}_{4}$ Tunnel | 20-Inch Mach 6 Tunnel ${ }^{a}$ |
| :---: | :---: | :---: |
| $p_{t, 1}, \mathrm{psi}$ | 1600 | 31 |
| $T_{t, 1},{ }^{\circ} \mathrm{R}$ | 1260 | 866 |
| $p_{2}, \mathrm{psi}$ | $8.05 \times 10^{-2}$ | $2.49 \times 10^{-2}$ |
| $T_{2},{ }^{\circ} \mathrm{R}$ | 386.5 | 113.1 |
| $M_{2}$ | 5.77 | 5.77 |
| $N_{\mathrm{Re}, 2}, \mathrm{ft}^{-1}$ | $5.76 \times 10^{5}$ | $6.38 \times 10^{5}$ |

${ }^{a}$ Data taken from reference 12.

The nozzle designs presented in this paper were performed on a preliminary basis and several items should be considered for further study. As discussed previously, no parametric study of maximum turning angle or initial Mach number distribution was made, and therefore these warrant examination. Since the numerical capability exists (ref. 11) to compute internal flow fields using the full Euler and NavierStokes equations, these calculations should obviously be considered for verification of any design. In addition, once a preliminary design contour is chosen, it would be possible to vary the reservoir conditions parametrically to study the effects of off-design conditions. Finally, no attempt has been made to design the settling chamber or the converging contour upstream of the throat section. The effects of those component designs can best be determined by solutions to the Euler and Navier-Stokes equations.

## Concluding Remarks

An analysis, modification, and preliminary design study was performed for the nozzle contour of the Langley Hypersonic $\mathrm{CF}_{4}$ Tunnel. Objectives of this work were, first, to study the origination and propagation of centerline disturbances measured in the existing tunnel; second, to determine the feasibility of modifying the present nozzle contour to eliminate the centerline disturbances; and, last, to redesign a nozzle contour for the $\mathrm{CF}_{4}$ Tunnel for different reservoir and exit conditions using the original design methodology.

A review of the procedure used in the original design of the $\mathrm{CF}_{4}$ Tunnel revealed that the present nozzle was contoured incorrectly for the design reservoir and exit conditions. In general, the diameter of the current nozzle was shown to be too large over the length of the tunnel. The formation of disturbances, which are focused on the nozzle centerline, was predicted for the nominal reservoir conditions, and the predicted disturbance pattern across the nozzle exit was in agreement with the measurements. A modified contour for the nozzle of the $\mathrm{CF}_{4}$ Tunnel was designed for the portion downstream of the maximum turning angle. This new contour theoretically provided a uniform exit flow for the nominal reservoir condition. New nozzle contours were designed for the $\mathrm{CF}_{4}$ Tunnel to provide uniform flow at the nozzle exit with a Mach number and Reynolds number combination which closely matches that attainable in the Langley 20-Inch Mach 6 Tunnel. Two nozzle contours were designed: one having the same exit radius but a larger mass flow rate than that of the existing $\mathrm{CF}_{4}$ Tunnel, and the other having the same mass flow rate but a smaller exit radius than that of the existing $\mathrm{CF}_{4}$ Tunnel.

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Table I. Design Wall Coordinates for the $\mathrm{CF}_{4}$ Nozzle
(a) $\dot{m}$ equal

| $x$, in. | $r$, in. |
| :---: | :---: |
| 0.000 | 0.2229 |
| .200 | .2240 |
| .400 | .2264 |
| .600 | .2299 |
| .800 | .2347 |
| 1.000 | .2406 |
| 1.200 | .2478 |
| 1.400 | .2558 |
| 1.600 | .2650 |
| 1.800 | .2761 |
| 2.000 | .2888 |
| 2.200 | .3032 |
| 2.400 | .3194 |
| 2.600 | .3376 |
| 2.800 | .3581 |
| 3.000 | .3811 |
| 3.200 | .4071 |
| 3.400 | .4366 |
| 3.600 | .4703 |
| 3.800 | .5083 |
| 4.000 | .5501 |
| 4.200 | .5950 |
| 4.400 | .6425 |
| 4.600 | .6921 |
| 4.800 | .7434 |
| 5.000 | .7962 |
| 5.200 | .8501 |
| 5.400 | .9050 |
| 5.600 | .9606 |
| 5.800 | 1.0169 |
| 6.000 | 1.0738 |
| 6.200 | 1.1311 |
| 6.400 | 1.1888 |
| 6.600 | 1.2408 |
| 6.800 | 1.3050 |
| 7.000 | 1.3635 |
| 7.200 | 1.4221 |
| 7.400 | 1.4808 |
| 7.600 | 1.5397 |
| 7.800 | 1.5987 |
| 8.000 | 1.6577 |
| 8.200 | 1.7168 |
| 8.400 | 1.7760 |
| 8.600 | 1.8353 |
| 3.800 | 1.8946 |
| 9.000 | 1.9539 |


| $x$, in. | $r$, in. |
| :---: | :---: |
| 9.200 | 2.0129 |
| 9.400 | 2.0716 |
| 9.600 | 2.1298 |
| 9.800 | 2.1875 |
| 10.000 | 2.2448 |
| 10.200 | 2.3015 |
| 10.400 | 2.3578 |
| 10.600 | 2.4136 |
| 10.800 | 2.4688 |
| 11.000 | 2.5236 |
| 11.200 | 2.5778 |
| 11.400 | 2.6316 |
| 11.600 | 2.6848 |
| 11.800 | 2.7376 |
| 12.000 | 2.7898 |
| 12.200 | 2.8416 |
| 12.400 | 2.8928 |
| 12.600 | 2.9436 |
| 12.800 | 2.9940 |
| 13.000 | 3.0438 |
| 13.200 | 3.0932 |
| 13.400 | 3.1420 |
| 13.600 | 3.1905 |
| 13.800 | 3.2384 |
| 14.000 | 3.2859 |
| 14.200 | 3.3329 |
| 14.400 | 3.3795 |
| 14.600 | 3.4257 |
| 14.800 | 3.4714 |
| 15.000 | 3.5167 |
| 15.200 | 3.5616 |
| 15.400 | 3.6060 |
| 15.600 | 3.6501 |
| 15.800 | 3.6937 |
| 16.000 | 3.7370 |
| 16.200 | 3.7798 |
| 16.400 | 3.8222 |
| 16.600 | 3.8643 |
| 16.800 | 3.9059 |
| 17.000 | 3.9472 |
| 17.200 | 3.9881 |
| 17.400 | 4.0286 |
| 17.600 | 4.0688 |
| 17.800 | 4.1080 |
| 18.000 | 4.1480 |
| 18.200 | 4.1871 |

Table I. Continued
(a) Continued

| $x$, in. | $r$, in. | $x$, in. | $r$, in. |
| :---: | :---: | :---: | :---: |
| 18.400 | 4.2258 | 27.600 | 5.6837 |
| 18.600 | 4.2642 | 27.800 | 5.7096 |
| 18.800 | 4.3023 | 28.000 | 5.7352 |
| 19.000 | 4.3399 | 28.200 | 5.7606 |
| 19.200 | 4.3772 | 28.400 | 5.7857 |
| 19.400 | 4.4142 | 28.600 | 5.8106 |
| 19.600 | 4.4508 | 28.800 | 5.8353 |
| 19.800 | 4.4871 | 29.000 | 5.8598 |
| 20.000 | 4.5231 | 29.200 | 5.8840 |
| 20.200 | 4.5587 | 29.400 | 5.9080 |
| 20.400 | 4.5940 | 29.600 | 5.9317 |
| 20.600 | 4.6290 | 29.800 | 5.9552 |
| 20.800 | 4.6637 | 30.000 | 5.9784 |
| 21.000 | 4.6982 | 30.500 | 6.0355 |
| 21.200 | 4.7322 | 31.000 | 6.0915 |
| 21.400 | 4.7660 | 31.500 | 6.1464 |
| 21.600 | 4.7995 | 32.000 | 6.2000 |
| 21.800 | 4.8327 | 32.500 | 0.2521 |
| 22.000 | 4.8656 | 33.000 | 6.3029 |
| 22.200 | 4.8982 | 33.500 | 6.3527 |
| 22.400 | 4.9305 | 34.000 | 6.4016 |
| 22.600 | 4.9626 | 34.500 | 6.4492 |
| 22.800 | 4.9944 | 35.000 | 6.4955 |
| 23.000 | 5.0259 | 35.500 | 6.5407 |
| 23.200 | 5.0571 | 36.000 | 6.5851 |
| 23.400 | 5.0880 | 36.500 | 6.6286 |
| 23.600 | 5.1186 | 37.000 | 6.6709 |
| 23.800 | 5.1490 | 37.500 | 6.7119 |
| 24.000 | 5.1790 | 38.000 | 6.7519 |
| 24.200 | 5.2088 | 38.500 | 0.7914 |
| 24.400 | 5.2384 | 39.000 | 6.8300 |
| 24.600 | 5.2677 | 39.500 | 0.8675 |
| 24.800 | 5.2968 | 40.000 | 6.9037 |
| 25.000 | 5.3258 | 40.500 | 0.9390 |
| 25.200 | 5.3546 | 41.000 | 6.9736 |
| 25.400 | 5.3832 | 41.500 | 7.0075 |
| 25.600 | 5.4117 | 42.000 | 7.0407 |
| 25.800 | 5.4399 | 42.500 | 7.0731 |
| 26.000 | 5.4679 | 43.000 | 7.1044 |
| 26.200 | 5.4957 | 43.500 | 7.1347 |
| 26.400 | 5.5233 | 44.000 | 7.1641 |
| 26.600 | 5.5507 | 44.500 | 7.1929 |
| 26.800 | 5.5778 | 45.000 | 7.2211 |
| 27.000 | 5.6047 | 45.500 | 7.2488 |
| 27.200 | 5.6313 | 46.000 | 7.2758 |
| 27.400 | 5.6576 | 46.500 | 7.3018 |

Table I. Continued
(a) Concluded

| $x$, in. | $r$, in. |
| :---: | :---: |
| 47.000 | 7.3267 |
| 47.500 | 7.3509 |
| 48.000 | 7.3744 |
| 48.500 | 7.3975 |
| 49.000 | 7.4200 |
| 49.500 | 7.4420 |
| 50.000 | 7.4633 |
| 50.500 | 7.4839 |
| 51.000 | 7.5039 |
| 51.500 | 7.5230 |
| 52.000 | 7.5413 |
| 32.500 | 7.5591 |
| 53.000 | 7.5764 |
| 53.500 | 7.5933 |
| 54.000 | 7.6096 |
| 54.500 | 7.6254 |
| 55.000 | 7.6408 |
| 55.500 | 7.6558 |
| 56.000 | 7.6701 |
| 56.500 | 7.6837 |
| 57.000 | 7.6966 |
| 57.500 | 7.7090 |
| 58.000 | 7.7209 |
| 58.500 | 7.7324 |


| $x$, in. | $r$, in. |
| :---: | :---: |
| 59.000 | 7.7436 |
| 59.500 | 7.7543 |
| 60.000 | 7.7646 |
| 60.500 | 7.7745 |
| 61.000 | 7.7840 |
| 61.500 | 7.7931 |
| 62.000 | 7.8018 |
| 62.500 | 7.8101 |
| 63.000 | 7.8181 |
| 63.500 | 7.8257 |
| 64.000 | 7.8329 |
| 64.500 | 7.8395 |
| 65.000 | 7.8457 |
| 65.500 | 7.8514 |
| 66.000 | 7.8568 |
| 66.500 | 7.8619 |
| 67.000 | 7.8668 |
| 67.500 | 7.8713 |
| 68.000 | 7.8755 |
| 68.500 | 7.8795 |
| 69.000 | 7.8832 |
| 69.500 | 7.8863 |
| 70.000 | 7.8889 |
| 70.425 | 7.8910 |

Table I. Continued
(b) $r_{e}$ equal

| $x$, in. | $r$, in. |
| :---: | :---: |
| 0.000 | 0.2792 |
| .200 | .2802 |
| .400 | .2822 |
| .600 | .2851 |
| .800 | .2890 |
| 1.000 | .2939 |
| 1.200 | .2997 |
| 1.400 | .3065 |
| 1.600 | .3141 |
| 1.800 | .3223 |
| 2.000 | .3316 |
| 2.200 | .3425 |
| 2.400 | .3547 |
| 2.000 | .3682 |
| 2.800 | .3830 |
| 3.000 | .3993 |
| 3.200 | .4173 |
| 3.400 | .4370 |
| 3.600 | .4586 |
| 3.800 | .4824 |
| 4.000 | .5086 |
| 4.200 | .5375 |
| 4.400 | .5697 |
| 4.600 | .6054 |
| 4.800 | .6445 |
| 5.000 | .6864 |
| 5.200 | .7309 |
| 5.400 | .7775 |
| 5.600 | .8260 |
| 5.800 | .8760 |
| 6.000 | .9274 |
| 6.200 | .9799 |
| 6.400 | 1.0334 |
| 6.600 | 1.0877 |
| 6.800 | 1.1427 |
| 7.000 | 1.1984 |
| 7.200 | 1.2545 |
| 7.400 | 1.3111 |
| 7.600 | 1.3682 |
| 7.800 | 1.4255 |
| 8.000 | 1.4831 |
| 8.200 | 1.5410 |
| 8.400 | 1.5991 |
| 8.600 | 1.6573 |
| 8.800 | 1.7158 |
| 9.000 | 1.7743 |
|  |  |


| $x$, in. | $r$, in. |
| :---: | :---: |
| 9.200 | 1.8330 |
| 9.400 | 1.8918 |
| 9.600 | 1.9506 |
| 9.800 | 2.0095 |
| 10.000 | 2.0685 |
| 10.200 | 2.1276 |
| 10.400 | 2.1866 |
| 10.600 | 2.2457 |
| 10.800 | 2.3048 |
| 11.000 | 2.3640 |
| 11.200 | 2.4233 |
| 11.400 | 2.4824 |
| 11.600 | 2.5413 |
| 11.800 | 2.5997 |
| 12.000 | 2.6579 |
| 12.200 | 2.7156 |
| 12.400 | 2.7730 |
| 12.600 | 2.8299 |
| 12.800 | 2.8865 |
| 13.000 | 2.9427 |
| 13.200 | 2.9985 |
| 13.400 | 3.0539 |
| 13.600 | 3.1089 |
| 13.800 | 3.1635 |
| 14.000 | 3.2178 |
| 14.200 | 3.2716 |
| 14.400 | 3.3250 |
| 14.600 | 3.3780 |
| 14.800 | 3.4305 |
| 15.000 | 3.4827 |
| 15.200 | 3.5345 |
| 15.400 | 3.5859 |
| 15.600 | 3.6369 |
| 15.800 | 3.6876 |
| 16.000 | 3.7378 |
| 16.200 | 3.7877 |
| 16.400 | 3.8372 |
| 16.600 | 3.8864 |
| 16.800 | 3.9351 |
| 17.000 | 3.9835 |
| 17.200 | 4.0315 |
| 17.400 | 4.0791 |
| 17.600 | 4.1264 |
| 17.800 | 4.1733 |
| 18.000 | 4.2198 |
| 18.200 | 4.2660 |
|  |  |

Table I. Continued
(b) Continued

| $x$, in. | $r$, in. |
| :---: | :---: |
| 18.400 | 4.3119 |
| 18.600 | 4.3575 |
| 18.800 | 4.4027 |
| 19.000 | 4.4475 |
| 19.200 | 4.4921 |
| 19.400 | 4.5363 |
| 19.600 | 4.5802 |
| 19.800 | 4.6237 |
| 20.000 | 4.6669 |
| 20.200 | 4.7098 |
| 20.400 | 4.7523 |
| 20.600 | 4.7946 |
| 20.800 | 4.8365 |
| 21.000 | 4.8781 |
| 21.200 | 4.9194 |
| 21.400 | 4.9604 |
| 21.600 | 5.0011 |
| 21.800 | 5.0415 |
| 22.000 | 5.0816 |
| 22.200 | 5.1214 |
| 22.400 | 5.1609 |
| 22.600 | 3.2002 |
| 22.800 | 5.2391 |
| 23.000 | 5.2778 |
| 23.200 | 5.3162 |
| 23.400 | 5.3543 |
| 23.600 | 5.3922 |
| 23.800 | 5.4297 |
| 24.000 | 5.4670 |
| 24.200 | 5.5040 |
| 24.400 | 5.5408 |
| 24.600 | 5.5773 |
| 24.800 | 5.6135 |
| 25.000 | 5.6494 |
| 25.200 | 5.6852 |
| 25.400 | 5.7206 |
| 25.600 | 5.7559 |
| 25.800 | 5.7908 |
| 20.000 | 5.8256 |
| 26.200 | 5.8601 |
| 26.400 | 5.8943 |
| 26.600 | 5.9283 |
| 26.800 | 5.9620 |
| 27.000 | 5.9955 |
| 27.200 | 6.0287 |
| 27.400 | 6.0617 |


| $x$, in. | $r$, in. |
| :---: | :---: |
| 27.600 | 6.0944 |
| 27.800 | 0.1269 |
| 28.000 | 6.1591 |
| 28.200 | 6.1912 |
| 28.400 | 6.2229 |
| 28.600 | 6.2545 |
| 28.800 | 6.2859 |
| 29.000 | 0.3170 |
| 29.200 | 6.3479 |
| 29.400 | 6.3786 |
| 29.600 | 6.4091 |
| 29.800 | 6.4394 |
| 30.000 | 6.4695 |
| 30.500 | 6.5438 |
| 31.000 | 6.6169 |
| 31.500 | 0.6889 |
| 32.000 | 6.7601 |
| 32.500 | 6.8303 |
| 33.000 | 6.8997 |
| 33.500 | 6.9680 |
| 34.000 | 7.0351 |
| 34.500 | 7.1007 |
| 35.000 | 7.1649 |
| 35.500 | 7.2278 |
| 36.000 | 7.2897 |
| 36.500 | 7.3506 |
| 37.000 | 7.4105 |
| 37.500 | 7.4690 |
| 38.000 | 7.5262 |
| 38.500 | 7.5824 |
| 39.000 | 7.6379 |
| 39.500 | 7.0925 |
| 40.000 | 7.7458 |
| 40.500 | 7.7979 |
| 41.000 | 7.8492 |
| 41.500 | 7.8996 |
| 42.000 | 7.9491 |
| 42.500 | 7.9978 |
| 43.000 | 8.0456 |
| 43.500 | 8.0922 |
| 44.000 | 8.1379 |
| 44.500 | 8.1829 |
| 45.000 | 8.2273 |
| 45.500 | 8.2709 |
| 46.000 | 8.3134 |
| 46.500 | 8.3550 |

Table I. Concluded
(b) Concluded

| $x$, in. | $r$, in. |
| :---: | :---: |
| 47.000 | 8.3956 |
| 47.500 | 8.4356 |
| 48.000 | 8.4751 |
| 48.500 | 8.5140 |
| 49.000 | 8.5521 |
| 49.500 | 8.5892 |
| 50.000 | 8.6253 |
| 50.500 | 8.6607 |
| 51.000 | 8.6955 |
| 51.500 | 8.7297 |
| 52.000 | 8.7634 |
| 52.500 | 8.7964 |
| 53.000 | 8.8288 |
| 53.500 | 8.8604 |
| 54.000 | 8.8913 |
| 54.500 | 8.9213 |
| 55.000 | 8.9506 |
| 55.500 | 8.9794 |
| 56.000 | 9.0077 |
| 56.500 | 9.0356 |
| 57.000 | 9.0630 |
| 57.500 | 9.0898 |
| 58.000 | 9.1158 |
| 58.500 | 9.1411 |
| 59.000 | 9.1657 |
| 59.500 | 9.1897 |
| 60.000 | 9.2132 |
| 60.500 | 9.2363 |
| 61.000 | 9.2589 |
| 61.500 | 9.2810 |
| 62.000 | 9.3027 |
| 62.500 | 9.3240 |
| 63.000 | 9.3448 |
| 63.500 | 9.3650 |
| 64.000 | 9.3844 |
| 64.500 | 9.4031 |
| 65.000 | 9.4214 |
| 65.500 | $y .4392$ |
| 66.000 | 9.4567 |
| 66.500 | 9.4737 |
| 67.000 | 9.4903 |
| 67.500 | 9.5005 |
|  |  |
|  |  |


| $x$, in. | $r$, in. |
| :---: | :---: |
| 68.000 | 9.5224 |
| 68.500 | 9.5378 |
| 69.000 | 9.5528 |
| 69.500 | $y .5675$ |
| 70.000 | 9.5818 |
| 70.500 | 9.5956 |
| 71.000 | 9.6087 |
| 71.500 | 9.6213 |
| 72.000 | 9.6334 |
| 72.500 | 9.6452 |
| 73.000 | 9.6568 |
| 73.500 | 9.6680 |
| 74.000 | 9.6788 |
| 74.500 | 9.6894 |
| 75.000 | 9.6995 |
| 75.500 | 9.7094 |
| 76.000 | 9.7190 |
| 76.500 | 9.7282 |
| 77.000 | 9.7371 |
| 77.500 | 9.7458 |
| 78.000 | 9.7541 |
| 78.500 | 9.7621 |
| 79.000 | 9.7698 |
| 79.500 | 9.7773 |
| 80.000 | 9.7844 |
| 80.500 | 9.7912 |
| 81.000 | 9.7973 |
| 81.500 | 9.8032 |
| 82.000 | 9.8087 |
| 82.500 | 9.8141 |
| 83.000 | 9.8192 |
| 83.500 | 9.8241 |
| 84.000 | 9.8287 |
| 84.500 | 9.8331 |
| 85.000 | 9.8373 |
| 85.500 | 9.8412 |
| 86.000 | 9.8448 |
| 86.500 | 9.8483 |
| 87.000 | 9.8516 |
| 87.500 | 9.8546 |
| 88.000 | 9.8570 |
| 88.219 | 9.8580 |
|  |  |
|  |  |


(a) Design method of characteristics.

(b) Analysis method of characteristics.

Figure 1. Diagram of supersonic wind-tunnel nozzle.


Figure 2. Comparison of existing and corrected $\mathrm{CF}_{4}$ nozzle contours. $M_{2}=6 ; p_{t, 1}=2500 \mathrm{psi} ; T_{t, 1}=1260^{\circ} \mathrm{R}$.


Figure 3. Exit radius as function of Mach number for one-dimensional inviscid flow of $\mathrm{CF}_{4}$ gas.


Figure 4. Theoretical and measured nozzle-wall temperature distributions.


Figure 5. Computational results for flow field in existing $\mathrm{CF}_{4}$ nozzle. $p_{t, 1}=1742 \mathrm{psi} ; T_{t, 1}=1274.3^{\circ} \mathrm{R}$.

(a) $p_{t, 1}=2552 \mathrm{psi} ; T_{t, 1}=1481.4^{\circ} \mathrm{R}$.

Figure 6. Comparison of computed and measured pitot pressure profiles across test section at nozzle exit.

(b) $p_{t, 1}=1742 \mathrm{psi} ; T_{t, 1}=1274.3^{\circ} \mathrm{R}$.

Figure 6. Continued.


Figure 6. Concluded.


Figure 7. Modified $\mathrm{CF}_{4}$ nozzle contour for $p_{t, 1}=1500 \mathrm{psi}, T_{t, 1}=1260^{\circ} \mathrm{R}$, and $M_{2}=6$.


Figure 8. Inviscid $\mathrm{CF}_{4}$ nozzle-exit radius as function of one-dimensional mass flow rate.


Figure 9. Theoretical wall temperature distributions for new $\mathrm{CF}_{4}$ nozzles. $T_{t, 1}=1260^{\circ} \mathrm{R}$.


Figure 10. New $\mathrm{CF}_{4}$ nozzle contours. $M_{2}=5.77 ; p_{t, 1}=1600 \mathrm{psi} ; T_{t, 1}=1260^{\circ} \mathrm{R}$.


Figure 11. New $\mathrm{CF}_{4}$ nozzle contours in throat region. $M_{2}=5.77 ; p_{t, 1}=1600 \mathrm{psi} ; T_{t, 1}=1260^{\circ} \mathrm{R}$.


