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MANUAL FOR PROGRAM PSTRESS: PEEL STRESS COMPUTATION

D. A. Barkey and R. C. Madan

Douglas Aircraft Company
Long Beach, CA 90846

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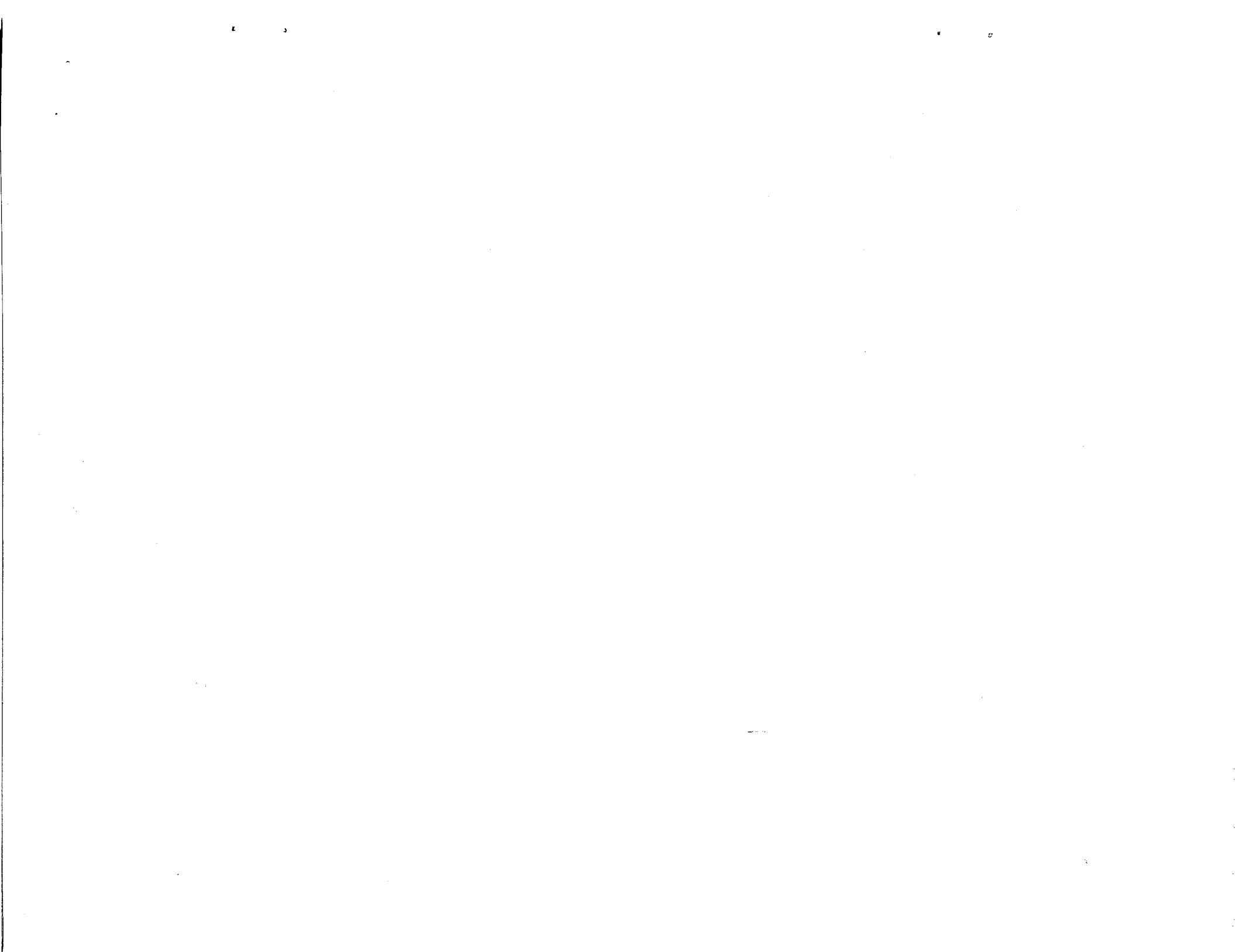
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**MANUAL FOR PROGRAM PSTRESS:
PEEL STRESS COMPUTATION**

Derek A. Barkey
Associate Engineer/Scientist

Ram C. Madan
Senior Engineer/Scientist

Douglas Aircraft Company
3855 Lakewood Blvd.
Long Beach, CA 90846

Nomenclature

- E_1, E_2 - Young's modulus of skin and stringer flange
 E_C - Young's modulus of adhesive
 D_1, D_2 - bending rigidity of skin and stringer flange
 G_C - shear modulus of adhesive
 t_1, t_2, t_C - thickness of skin, flange, and adhesive
 L - flange width
 P_1, P_2 - vertical shear force applied to skin and flange
 M_1, M_2 - moment applied to skin and flange
 F_1, F_2 - inplane force applied to skin and flange
 T_1, T_2 - $(t_1+t_C)/D_1, (t_2+t_C)/D_2$
 u_1, u_2 - inplane displacement of skin and flange
 w_1, w_2 - vertical deflection of skin and flange
 $U(i, j)$ - eigenvector
 $\lambda(i)$ - eigenvalue (LAM(i) in program output)
 $\mu(i)$ - $\sqrt{\lambda(i)}$ (MU(i) in program output)
 $C(i)$ - coefficient
 σ_1, σ_2 - inplane stress in skin and flange (SIG1, SIG2 in program output)
 σ_C - peel stress in adhesive (SIGC in program output)
 τ_C - shear stress in adhesive (TAUC in program output)

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1. INTRODUCTION

Computer program PSTRESS, which carries out an analysis of two bonded plates, was developed as part of NASA contract NAS1-17970. The goal of the NASA project is the development of key technology for the application of composite materials to high aspect ratio transport wing structure. The objectives of this effort are the design, manufacture, and test of cover panel structure for a baseline composite wing that will meet all strength, stiffness, aeroelastic, and damage tolerance requirements with maximum weight savings and minimum cost.

Test results have shown that impact in the area of a T-shaped stringer bonded to a composite skin can result in a debond. PSTRESS is a computer program developed to investigate the influence of various parameters on the adhesive stresses in such a bonded composite panel. It is an interactive program written in the FORTRAN 77 language. Input includes properties of the skin and stringer flange, loading on the skin and stringer flange, and properties of the adhesive. Output includes functions for the calculation of vertical and inplane displacements, inplane stress in the skin and stringer flange, and peel and shear stresses in the adhesive layer. These quantities are calculated at twenty points across the width of the flange. These numbers are presented in the output in tabular form.

2. ANALYTICAL DEVELOPMENT

Figure 1 shows the model used in the PSTRESS analysis. Subscripts 1, 2, and c refer to properties of the skin, stringer flange, and adhesive, respectively. A complete description of the analysis can be found in Reference 1. Briefly, a system of two fourth order and two second order differential equations with associated boundary equations are obtained using the principle of minimum potential energy. PSTRESS solves for the eigenvalues and eigenvectors of the system of equations and applies the boundary conditions to solve for the coefficients of functions for w_1 , w_2 , u_1 , and u_2 . Beam bending, linear elastic behavior of the material, and symmetry about the plane through the blade of the stringer are assumed throughout the analysis.

Peel stress and shear stress in the adhesive are determined from the vertical deflections (w_1 , w_2) and inplane displacements (u_1 , u_2). Equations 1a and 1b show the relations for these stresses in terms of u_1 , u_2 , w_1 , and w_2 .

$$\tau_c = G_c(2(u_1 - u_2) + D_1 T_1 dw_1/dx + D_2 T_2 dw_2/dx) / 2t_c \quad (1a)$$

$$\sigma_c = E_c(w_1 - w_2) / t_c \quad (1b)$$

PSTRESS calculates eigenvalues ($\lambda(i)$'s) and eigenvectors ($U(ij)$'s). Boundary conditions are then applied and the necessary coefficients of the solutions are determined. Figure 2 shows a simplified flow chart of the program.

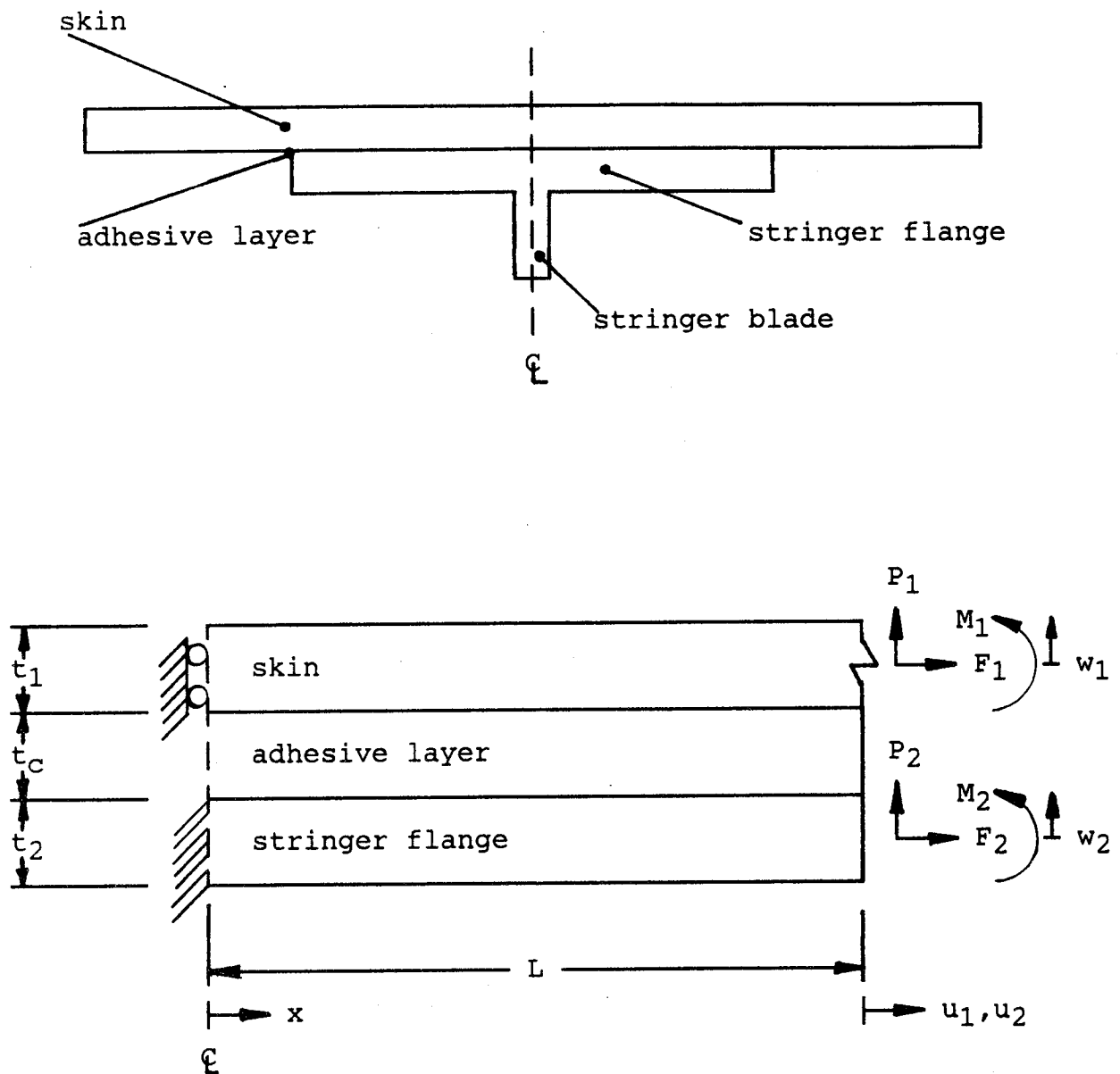


Figure 1
 Model of Skin/Stringer
 for PSTRESS Analysis
 (Not to Scale)

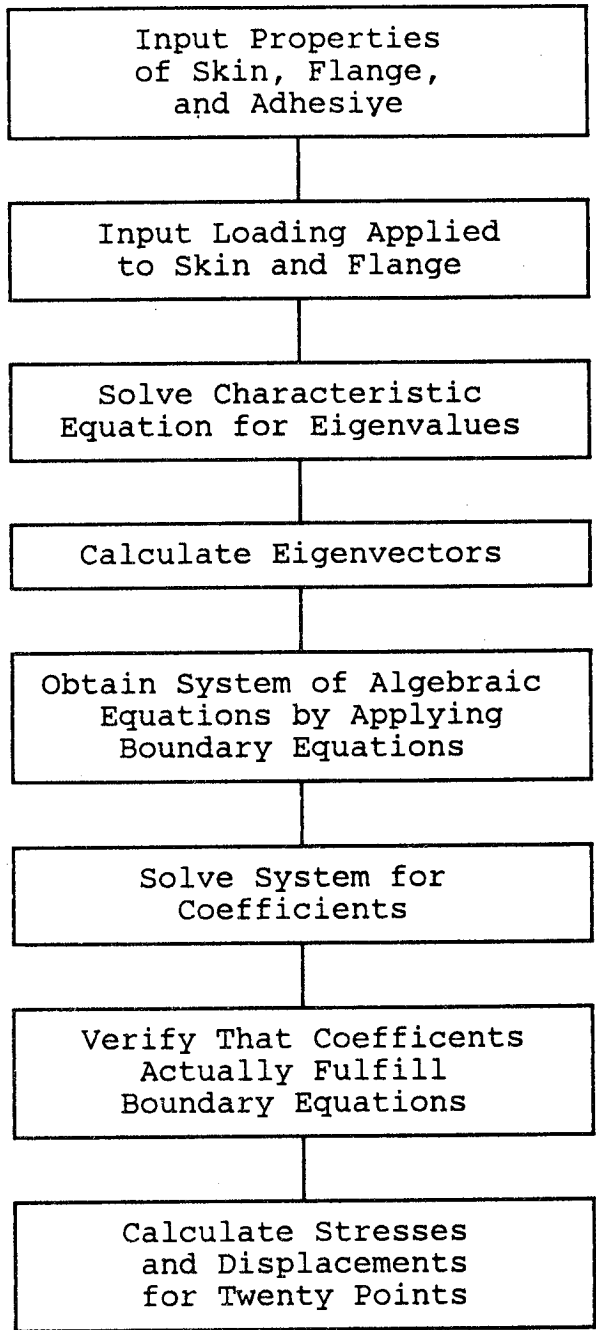


Figure 2
Flow Chart of PSTRESS

The solutions for the system of differential equations take the following form:

Case 1

Inplane force (F_1 or F_2) applied.

$$w_1 = [C(1)e^{\mu(1)x} + C(2)e^{-\mu(1)x} + C(3)e^{\mu(2)x} + C(4)e^{-\mu(2)x} + C(5)e^{\mu(3)x} + C(6)e^{-\mu(3)x} + C(7)e^{\mu(4)x} + C(8)e^{-\mu(4)x} + C(9)x + C(10)] / D_1$$

$$w_2 = [U(21)\{C(1)e^{\mu(1)x} + C(2)e^{-\mu(1)x}\} + U(22)\{C(3)e^{\mu(2)x} + C(4)e^{-\mu(2)x}\} + U(23)\{C(5)e^{\mu(3)x} + C(6)e^{-\mu(3)x}\} + U(24)\{C(7)e^{\mu(4)x} + C(8)e^{-\mu(4)x}\}] / D_2 + [C(9)x + C(10)] / D_1$$

$$u_1 = [U(51)\{C(1)e^{\mu(1)x} - C(2)e^{-\mu(1)x}\} / \mu(1) + U(52)\{C(3)e^{\mu(2)x} - C(4)e^{-\mu(2)x}\} / \mu(2) + U(53)\{C(5)e^{\mu(3)x} - C(6)e^{-\mu(3)x}\} / \mu(3) + U(54)\{C(7)e^{\mu(4)x} - C(8)e^{-\mu(4)x}\} / \mu(4) + C(11)x + C(13)] / E_1 t_1$$

$$u_2 = [-U(51)\{C(1)e^{\mu(1)x} - C(2)e^{-\mu(1)x}\} / \mu(1) - U(52)\{C(3)e^{\mu(2)x} - C(4)e^{-\mu(2)x}\} / \mu(2) - U(53)\{C(5)e^{\mu(3)x} - C(6)e^{-\mu(3)x}\} / \mu(3) - U(54)\{C(7)e^{\mu(4)x} - C(8)e^{-\mu(4)x}\} / \mu(4) + C(12)x + C(14)] / E_2 t_2$$

(2a-d)

Case 2

Inplane force not applied ($F_1 = F_2 = 0$).

$$w_1 = [C(1)e^{\mu(1)x} + C(2)e^{-\mu(1)x} + C(3)e^{\mu(2)x} + C(4)e^{-\mu(2)x} + C(5)e^{\mu(3)x} + C(6)e^{-\mu(3)x} + C(7)x^3 + C(8)x^2 + C(9)x + C(10)] / D_1$$

$$w_2 = [U(21)\{C(1)e^{\mu(1)x} + C(2)e^{-\mu(1)x}\} + U(22)\{C(3)e^{\mu(2)x} + C(4)e^{-\mu(2)x}\} + U(23)\{C(5)e^{\mu(3)x} + C(6)e^{-\mu(3)x}\}] / D_2 + [C(7)x^3 + C(8)x^2 + C(9)x + C(10)] / D_1$$

$$\begin{aligned}
u_1 = & [U(51)\{C(1)e^{\mu(1)x} - C(2)e^{-\mu(1)x}\}/\mu(1) + U(52)\{C(3)e^{\mu(2)x} - \\
& C(4)e^{-\mu(2)x}\}/\mu(2) + U(53)\{C(5)e^{\mu(3)x} - C(6)e^{-\mu(3)x}\}/\mu(3) - \\
& 3E_1t_1E_2t_2(T_1+T_2D_2/D_1)C(7)x^2 / (E_1t_1+E_2t_2) + C(11)x + C(13)] / E_1t_1 \\
u_2 = & [-U(51)\{C(1)e^{\mu(1)x} - C(2)e^{-\mu(1)x}\}/\mu(1) - U(52)\{C(3)e^{\mu(2)x} - \\
& C(4)e^{-\mu(2)x}\}/\mu(2) - U(53)\{C(5)e^{\mu(3)x} - C(6)e^{-\mu(3)x}\}/\mu(3) + \\
& 3E_1t_1E_2t_2(T_1+T_2D_2/D_1)C(7)x^2 / (E_1t_1+E_2t_2) + C(12)x + C(14)] / E_2t_2
\end{aligned}$$

(3a-d)

3. PROGRAM INPUT AND OUTPUT

Program Input

The operator is prompted for all input. Necessary input for the program is as follows:

- E_1 , D_1 , and t_1 : modulus, bending rigidity, and thickness of the skin
- E_2 , D_2 , and t_2 : modulus, bending rigidity, and thickness of the stringer flange
- E_C , G_C , and t_C : extensional modulus, shear modulus, and thickness of the adhesive
- L : width of stringer flange
- P_1 , M_1 , and F_1 : running vertical shear force, running moment, and running inplane force applied to the skin
- P_2 , M_2 , and F_2 : running vertical shear force, running moment, and running inplane force applied to the stringer flange

Program Output

All output is displayed to the screen and is also written to a file entitled "PSTRESS OUTPUT". The following quantities are output:

- Echo of input quantities
- Eigenvalues and their square roots: $\lambda(i)$'s and $\mu(i)$'s
- Eigenvectors: w_1 ($U(1,j)$), w_2 ($U(2,j)$), d^2w_1/dx^2 ($U(3,j)$), d^2w_2/dx^2 ($U(4,j)$), and u_1 ($U(5,j)$).
- Coefficients: $C(i)$'s
- Deflections: w_1 , w_2 , u_1 , and u_2 for twenty points across flange width
- Stresses: σ_C , τ_C , σ_1 , and σ_2 for twenty points across flange width

4. SAMPLE PROBLEMS

The following pages contain the input and output for analyses using the data given below:

$E_1=4713$ ksi (32.473 GPa)	$E_2=4713$ ksi (32.473 GPa)
$D_1=3685$ lb-in (416 N-m)	$D_2=357$ lb-in (40.3 N-m)
$t_1=.2232$ in (5.7 mm)	$t_2=.1116$ in (2.8 mm)
$E_c=500$ ksi (3.445 GPa)	$G_c=45$ ksi (.310 GPa)
$t_c=.005$ in (.127 mm)	$L=1.2$ in (30.48 mm)

Case 1

$P_1=100$ lbs/in (17.5 kN/m)	$P_2=0$
$M_1=80$ lb-in/in (355.6 N-m/m)	$M_2=0$
$F_1=1600$ lbs/in (280 kN/m)	$F_2=0$

Case 2

$P_1=100$ lbs/in (17.5 kN/m)	$P_2=0$
$M_1=80$ lb-in/in (355.6 N-m/m)	$M_2=0$
$F_1=0$	$F_2=0$

The output is plotted for each case and shown in Figures 3 through 11. For comparison, the results of a NASTRAN finite element analysis of an equivalent model are also shown in figures 3, 4, 8, and 9.

pstress

P E E L S T R E S S

INPUT PROPERTIES OF SKIN: E1, D1, T1

?

4713000,3685,.2232

INPUT PROPERTIES OF FLANGE: E2, D2, T2

?

4713000,357,.1116

INPUT ADHESIVE PROPERTIES: EC, GC, TC

?

500000,45000,.005

INPUT FLANGE WIDTH: L

?

1.2

INPUT SHEAR, MOMENT, AND FORCE APPLIED TO SKIN

?

100,80,1600

INPUT SHEAR, MOMENT, AND FORCE APPLIED TO FLANGE

?

0,0,0

DO YOU WANT TO SEE EIGENVALUES, EIGENVECTORS, AND COEFFICIENTS(Y/N)?
"NO" IS DEFAULT.

n

X	DEFLECTIONS			
	W1	W2	U1	U2
0.0000	1.367E-05	1.381E-34	1.050E-35	8.999E-36
0.0188	1.508E-05	2.555E-06	1.211E-05	3.281E-05
0.0375	1.932E-05	9.278E-06	2.426E-05	6.555E-05
0.0750	3.609E-05	3.147E-05	4.875E-05	1.307E-04
0.1125	6.347E-05	6.247E-05	7.356E-05	1.951E-04
0.1500	1.008E-04	1.014E-04	9.875E-05	2.588E-04
0.2250	2.026E-04	2.033E-04	1.504E-04	3.836E-04
0.3000	3.369E-04	3.371E-04	2.039E-04	5.047E-04
0.4200	6.106E-04	6.106E-04	2.938E-04	6.900E-04
0.5400	9.457E-04	9.458E-04	3.891E-04	8.644E-04
0.6600	1.331E-03	1.331E-03	4.900E-04	1.028E-03
0.7800	1.756E-03	1.757E-03	5.967E-04	1.179E-03
0.9000	2.215E-03	2.216E-03	7.101E-04	1.318E-03
0.9750	2.518E-03	2.522E-03	7.855E-04	1.395E-03
1.0500	2.839E-03	2.853E-03	8.663E-04	1.462E-03

1.0875	3.011E-03	3.027E-03	9.097E-04	1.489E-03
1.1250	3.194E-03	3.203E-03	9.560E-04	1.510E-03
1.1625	3.394E-03	3.371E-03	1.006E-03	1.525E-03
1.1812	3.503E-03	3.450E-03	1.032E-03	1.529E-03
1.2000	3.618E-03	3.526E-03	1.060E-03	1.530E-03

X	STRESSES			
	SIGC	TAUC	SIG1	SIG2
0.0000	1.367E+03	-1.454E-28	3.044E+03	8.250E+03
0.0188	1.253E+03	4.787E+01	3.048E+03	8.240E+03
0.0375	1.004E+03	9.478E+01	3.060E+03	8.217E+03
0.0750	4.615E+02	1.806E+02	3.096E+03	8.145E+03
0.1125	1.003E+02	2.497E+02	3.141E+03	8.055E+03
0.1500	-5.543E+01	3.002E+02	3.191E+03	7.955E+03
0.2250	-6.465E+01	3.573E+02	3.303E+03	7.730E+03
0.3000	-1.612E+01	3.815E+02	3.427E+03	7.483E+03
0.4200	-7.243E-01	3.960E+02	3.636E+03	7.065E+03
0.5400	-2.991E+00	4.040E+02	3.851E+03	6.634E+03
0.6600	-1.139E+01	4.216E+02	4.072E+03	6.192E+03
0.7800	-2.808E+01	4.791E+02	4.312E+03	5.713E+03
0.9000	-7.832E+01	6.626E+02	4.611E+03	5.115E+03
0.9750	-4.089E+02	9.444E+02	4.883E+03	4.571E+03
1.0500	-1.345E+03	1.551E+03	5.311E+03	3.715E+03
1.0875	-1.650E+03	2.095E+03	5.621E+03	3.094E+03
1.1250	-8.826E+02	2.903E+03	6.019E+03	2.299E+03
1.1625	2.340E+03	4.061E+03	6.523E+03	1.292E+03
1.1812	5.308E+03	4.792E+03	6.824E+03	6.890E+02
1.2000	9.243E+03	5.629E+03	7.168E+03	1.272E-28

ANOTHER RUN WITH DIFFERENT LOADS(Y/N)?

Y

INPUT SHEAR, MOMENT, AND FORCE APPLIED TO SKIN

?

100,80,0

INPUT SHEAR, MOMENT, AND FORCE APPLIED TO FLANGE

?

0,0,0

DO YOU WANT TO SEE EIGENVALUES, EIGENVECTORS, AND COEFFICIENTS(Y/N)?

"NO" IS DEFAULT.

Y

EIGENVALUES AND THEIR SQUARE ROOTS:

LAM(1) 2.4945642E+01 +5.4952145E+02i

MU(1) 1.6956312E+01 +1.6204038E+01i

LAM(2) 2.4945642E+01 -5.4952145E+02i

MU(2) 1.6956312E+01 -1.6204038E+01i

LAM(3) 9.3258122E+01 +0.0000000E+00i
 MU(3) 9.6570245E+00 +0.0000000E+00i
 LAM(4) 0.0000000E+00 +0.0000000E+00i
 MU(4) 0.0000000E+00 +0.0000000E+00i

EIGENVECTORS:

U(1,1) 1.0000000E+00 +0.0000000E+00i
 U(1,2) 1.0000000E+00 +0.0000000E+00i
 U(1,3) 1.0000000E+00 +0.0000000E+00i
 U(1,4) 0.0000000E+00 +0.0000000E+00i
 U(2,1) -8.5764511E-01 +3.0784698E-01i
 U(2,2) -8.5764511E-01 -3.0784698E-01i
 U(2,3) 1.0521675E-01 +0.0000000E+00i
 U(2,4) 0.0000000E+00 +0.0000000E+00i
 U(3,1) 2.4945642E+01 +5.4952145E+02i
 U(3,2) 2.4945642E+01 -5.4952145E+02i
 U(3,3) 9.3258122E+01 +0.0000000E+00i
 U(3,4) 0.0000000E+00 +0.0000000E+00i
 U(4,1) -1.9056303E+02 -4.6361495E+02i
 U(4,2) -1.9056303E+02 +4.6361495E+02i
 U(4,3) 9.8123162E+00 +0.0000000E+00i
 U(4,4) 0.0000000E+00 +0.0000000E+00i
 U(5,1) -9.6065846E+02 +4.9828957E+02i
 U(5,2) -9.6065846E+02 -4.9828957E+02i
 U(5,3) 5.9785653E+02 +0.0000000E+00i
 U(5,4) 0.0000000E+00 +0.0000000E+00i

COEFFICIENTS ARE:

C(1) 2.0393568E-11 -1.6118227E-11i
 C(2) 1.7996557E-03 +3.4075299E-03i
 C(3) 2.0393568E-11 +1.6118227E-11i
 C(4) 1.7996557E-03 -3.4075299E-03i
 C(5) 5.6962153E-06 -4.3274595E-41i
 C(6) 7.9183185E-02 +2.1439069E-34i
 C(7) -4.2461834E+00 -8.9683208E-33i
 C(8) 2.5477184E+01 +3.2920959E-32i
 C(9) 7.1521850E-01 +1.3226020E-33i
 C(10) -3.2484452E-02 -1.4787472E-34i
 C(11) -8.3589469E+02 -1.0801215E-30i
 C(12) 8.3589469E+02 +1.0801215E-30i
 C(13) 4.5505142E+00 +1.2695253E-32i
 C(14) -4.5505142E+00 -1.2695253E-32i

DEFLECTIONS					
X	W1	W2	U1	U2	
0.0000	1.365E-05	-2.828E-36	1.494E-37	-4.687E-38	
0.0188	1.626E-05	3.748E-06	-1.417E-05	2.834E-05	
0.0375	2.407E-05	1.405E-05	-2.831E-05	5.662E-05	
0.0750	5.514E-05	5.056E-05	-5.639E-05	1.128E-04	
0.1125	1.064E-04	1.054E-04	-8.415E-05	1.683E-04	
0.1500	1.771E-04	1.776E-04	-1.115E-04	2.230E-04	
0.2250	3.742E-04	3.748E-04	-1.649E-04	3.299E-04	
0.3000	6.417E-04	6.419E-04	-2.164E-04	4.329E-04	
0.4200	1.207E-03	1.207E-03	-2.944E-04	5.889E-04	
0.5400	1.931E-03	1.931E-03	-3.668E-04	7.336E-04	
0.6600	2.801E-03	2.801E-03	-4.334E-04	8.668E-04	
0.7800	3.805E-03	3.805E-03	-4.940E-04	9.880E-04	
0.9000	4.935E-03	4.936E-03	-5.479E-04	1.096E-03	
0.9750	5.704E-03	5.707E-03	-5.775E-04	1.155E-03	
1.0500	6.522E-03	6.533E-03	-6.024E-04	1.205E-03	
1.0875	6.953E-03	6.966E-03	-6.125E-04	1.225E-03	
1.1250	7.401E-03	7.408E-03	-6.203E-04	1.241E-03	
1.1625	7.870E-03	7.851E-03	-6.255E-04	1.251E-03	
1.1812	8.114E-03	8.070E-03	-6.269E-04	1.254E-03	
1.2000	8.365E-03	8.287E-03	-6.274E-04	1.255E-03	

STRESSES					
X	SIGC	TAUC	SIG1	SIG2	
0.0000	1.365E+03	7.889E-31	-3.564E+03	7.127E+03	
0.0188	1.251E+03	4.881E+01	-3.559E+03	7.118E+03	
0.0375	1.002E+03	9.664E+01	-3.547E+03	7.094E+03	
0.0750	4.589E+02	1.843E+02	-3.510E+03	7.021E+03	
0.1125	9.747E+01	2.551E+02	-3.465E+03	6.929E+03	
0.1500	-5.827E+01	3.073E+02	-3.414E+03	6.827E+03	
0.2250	-6.717E+01	3.675E+02	-3.299E+03	6.597E+03	
0.3000	-1.826E+01	3.943E+02	-3.171E+03	6.342E+03	
0.4200	-2.261E+00	4.120E+02	-2.954E+03	5.908E+03	
0.5400	-3.469E+00	4.214E+02	-2.730E+03	5.460E+03	
0.6600	-9.155E+00	4.361E+02	-2.500E+03	5.000E+03	
0.7800	-1.996E+01	4.782E+02	-2.256E+03	4.512E+03	
0.9000	-4.764E+01	6.082E+02	-1.970E+03	3.939E+03	
0.9750	-3.025E+02	8.056E+02	-1.730E+03	3.460E+03	
1.0500	-1.084E+03	1.232E+03	-1.379E+03	2.758E+03	
1.0875	-1.371E+03	1.619E+03	-1.135E+03	2.270E+03	
1.1250	-7.798E+02	2.197E+03	-8.315E+02	1.663E+03	
1.1625	1.901E+03	3.030E+03	-4.593E+02	9.187E+02	
1.1812	4.435E+03	3.556E+03	-2.428E+02	4.856E+02	
1.2000	7.860E+03	4.153E+03	2.631E-03	-5.261E-03	

ANOTHER RUN WITH DIFFERENT LOADS(Y/N)?

n

SEND RESULTS TO PRINTER(Y/N)?

n

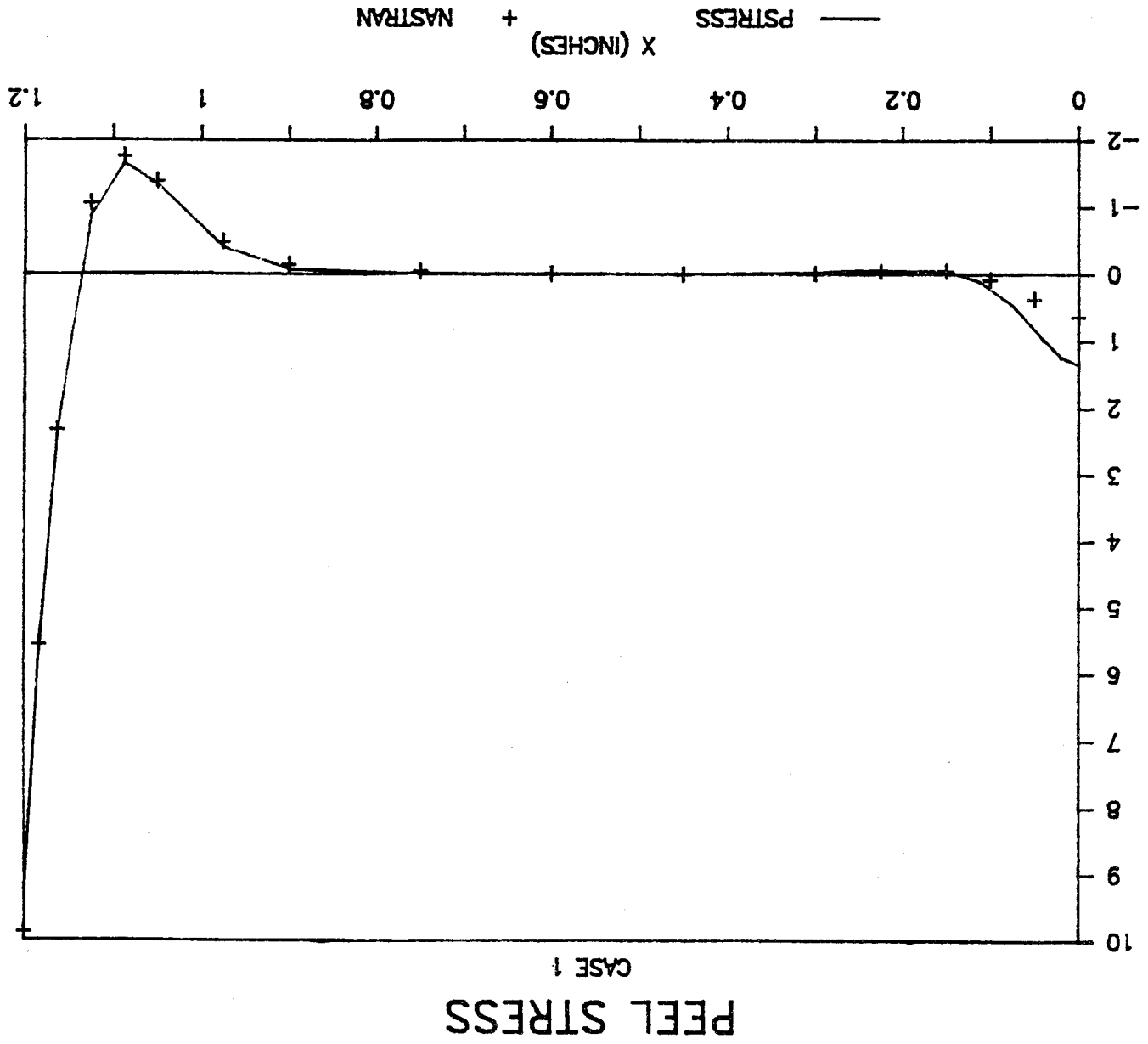


Figure 3. Peel stress in adhesive, Case 1.

PEEL STRESS CASE 2

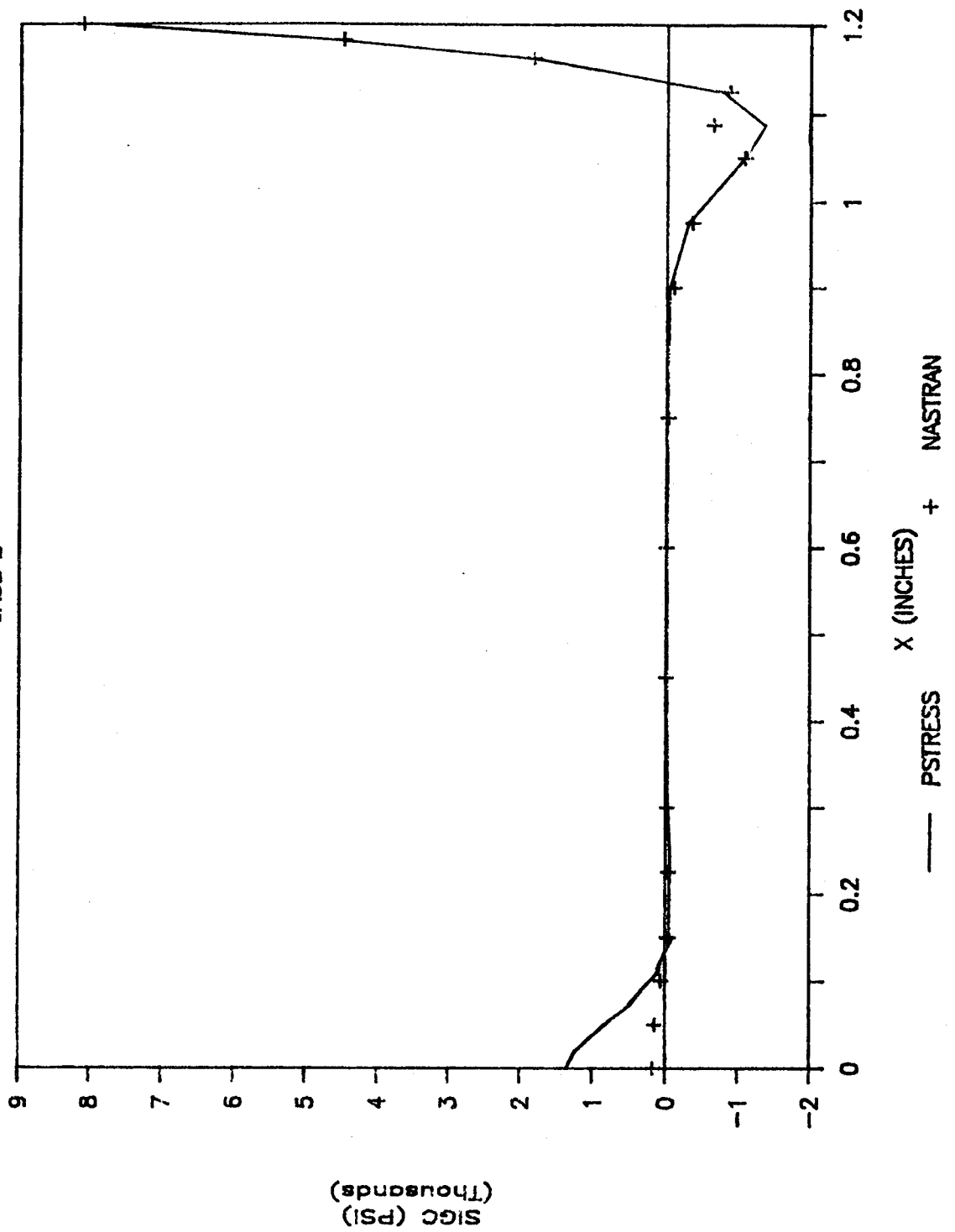


Figure 4. Peel stress in adhesive, Case 2.

ADHESIVE SHEAR STRESS

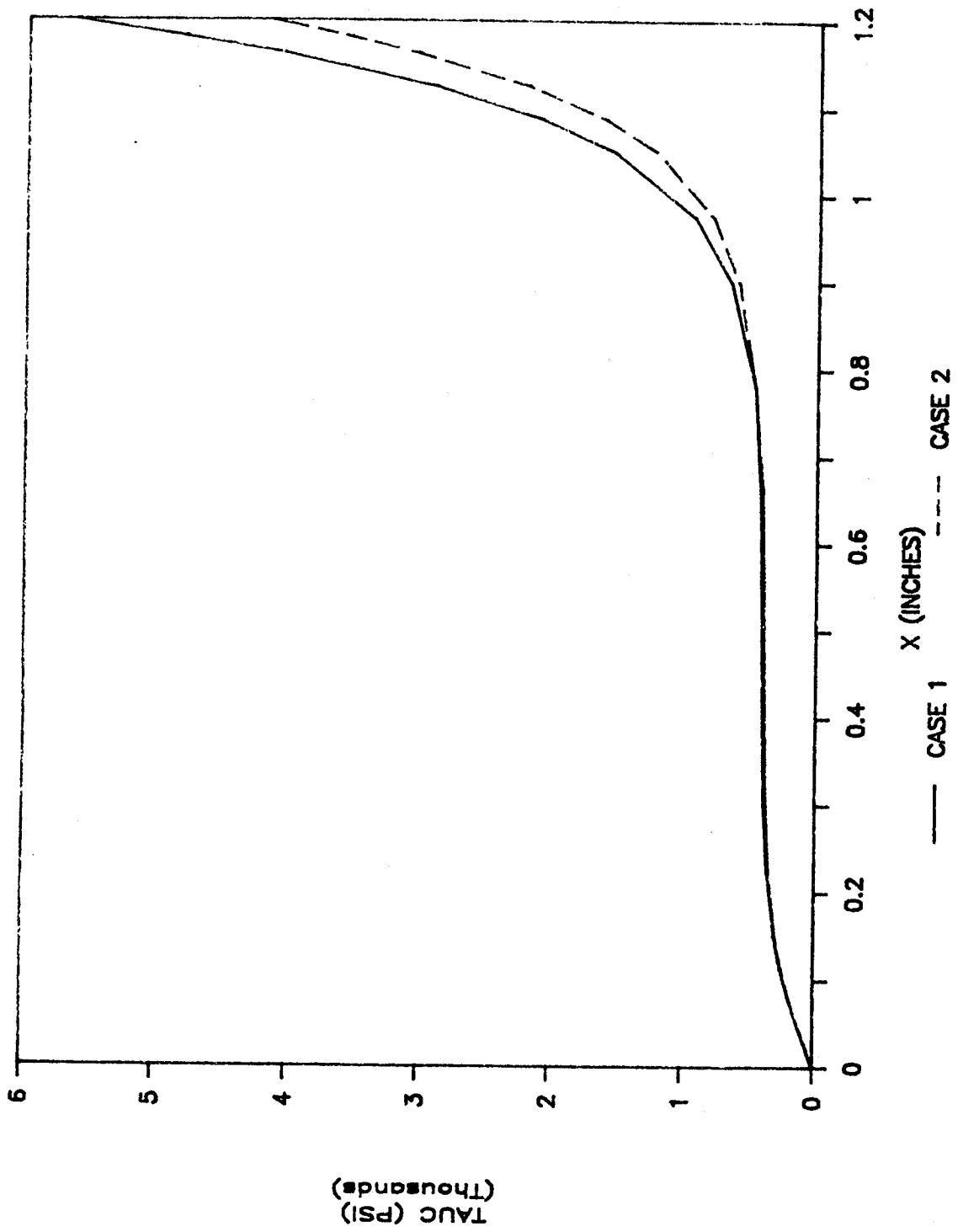


Figure 5. Shear stress in adhesive.

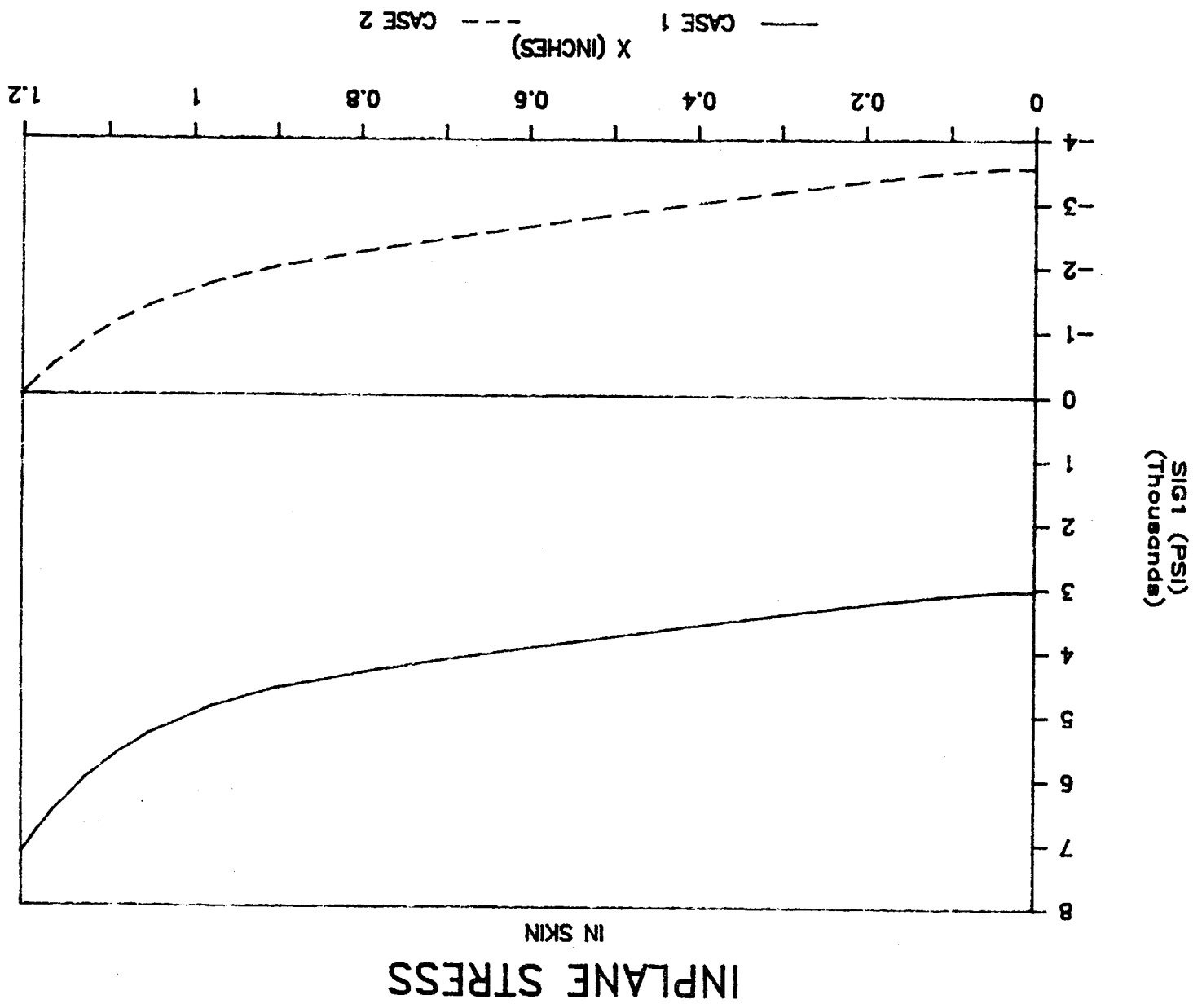


Figure 6. Inplane stress in skin laminate.

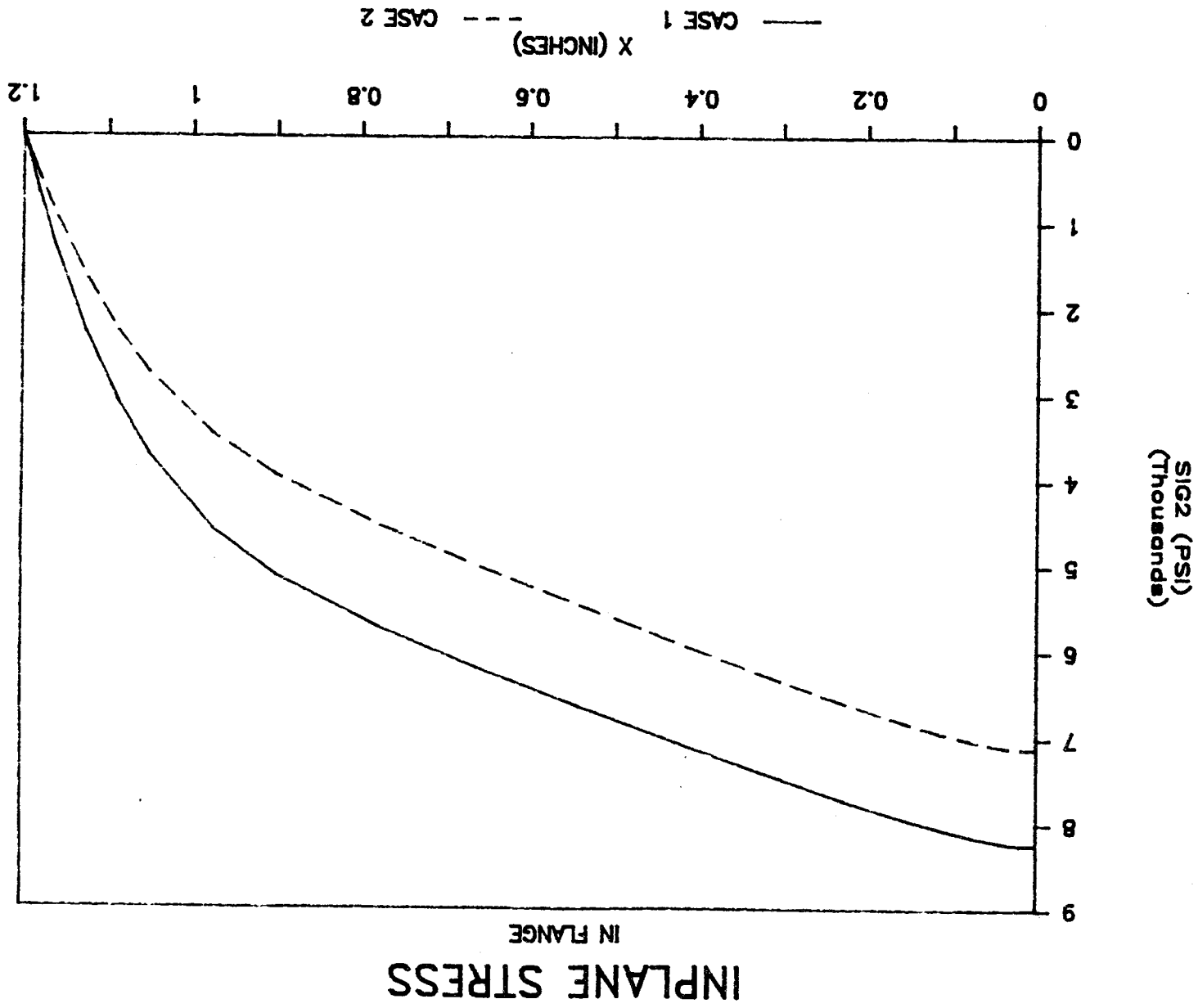


Figure 7. Inplane stress in flange laminate.

VERTICAL DEFLECTION OF SKIN

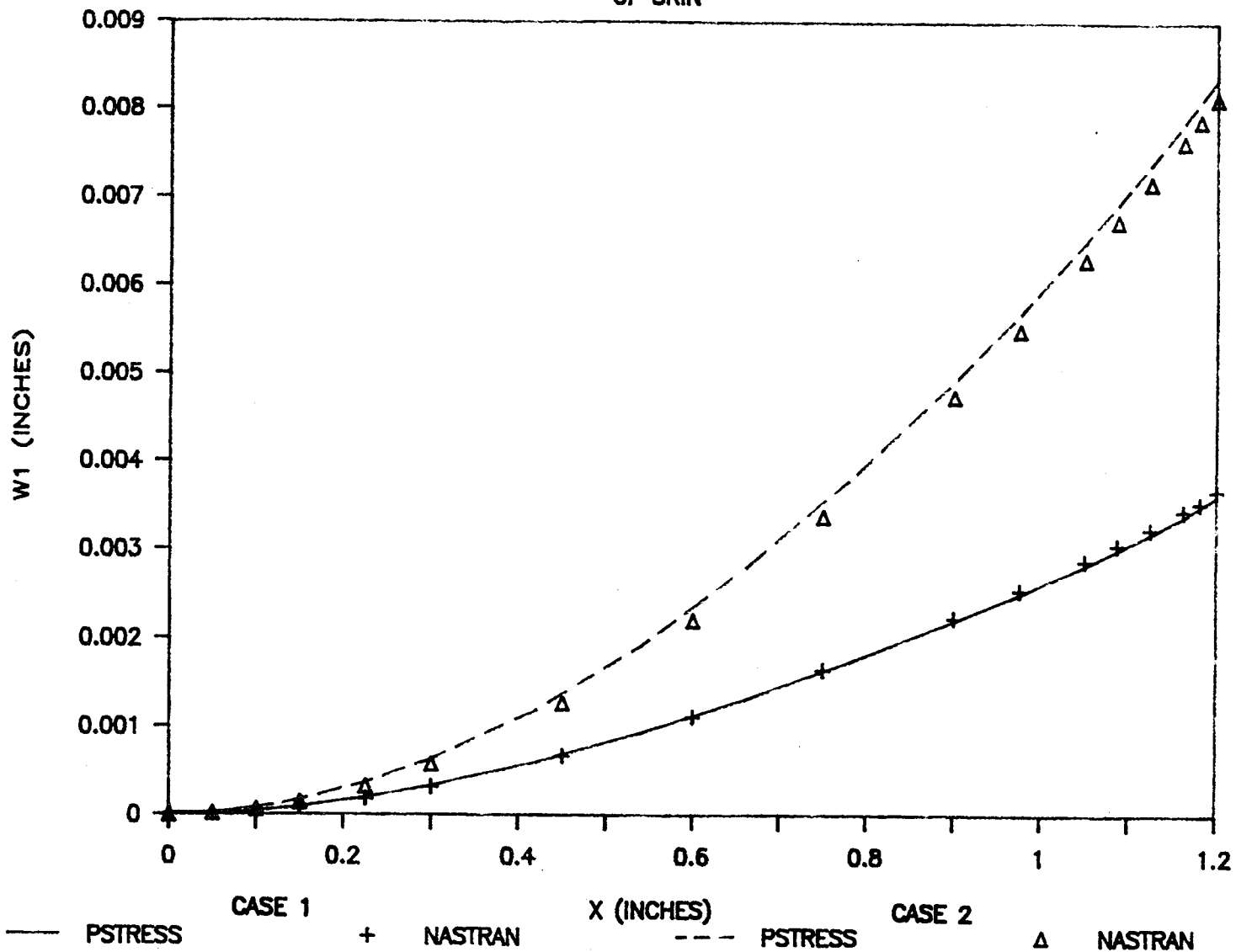


Figure 8. Out-of-plane displacement of skin laminate.

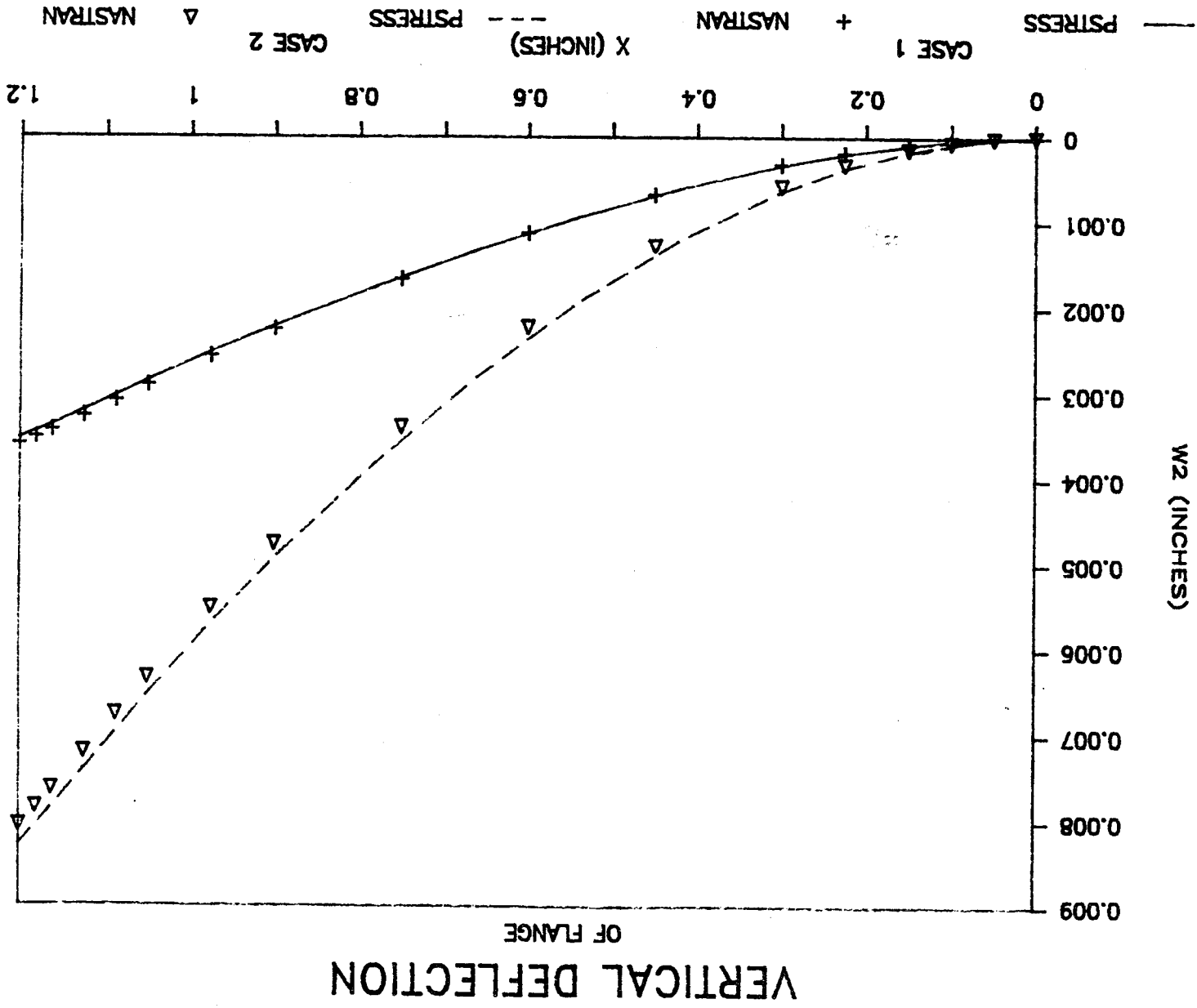


Figure 9. Out-of-plane displacement of flange laminate.

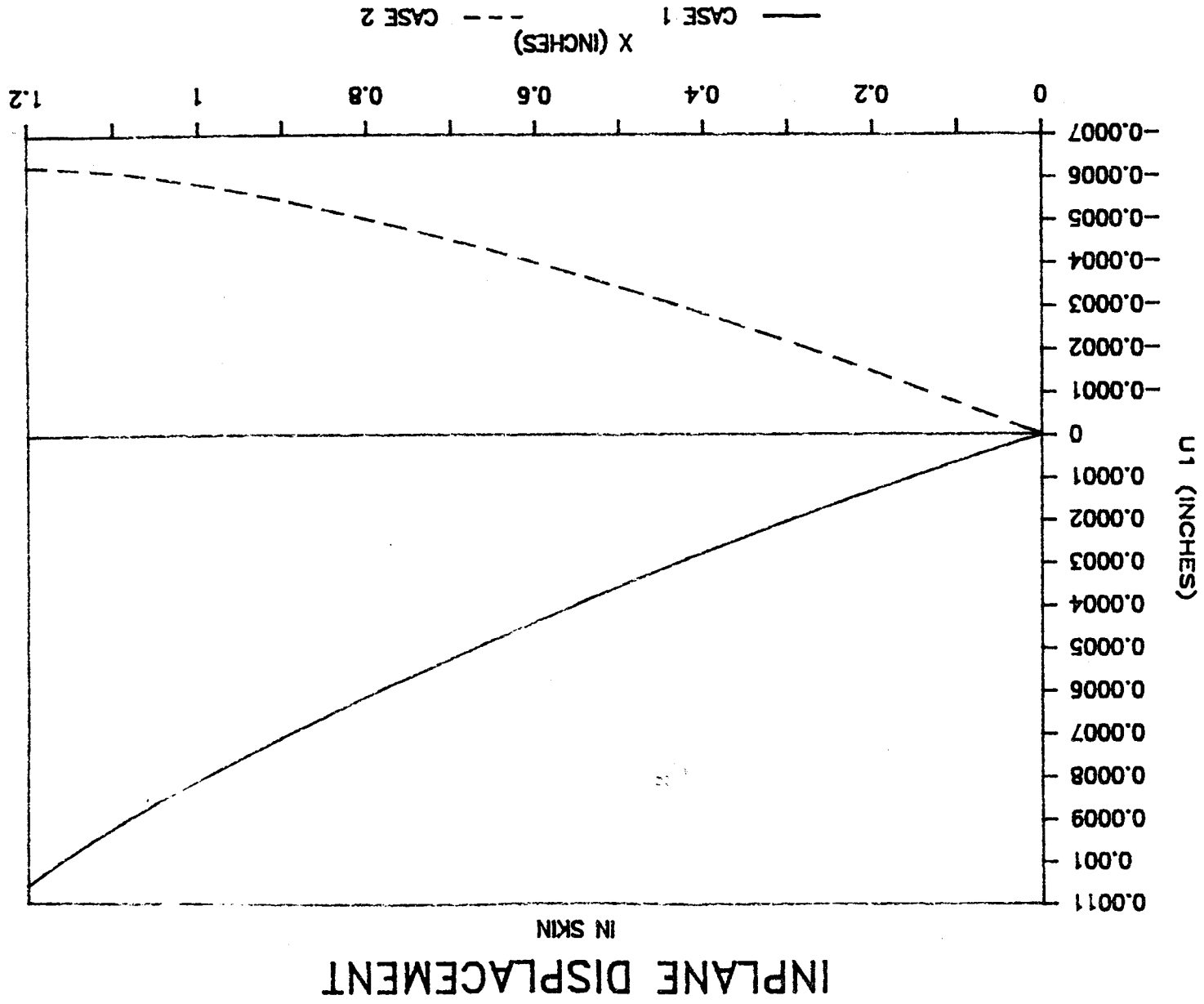


Figure 10. Inplane displacement of skin laminate.

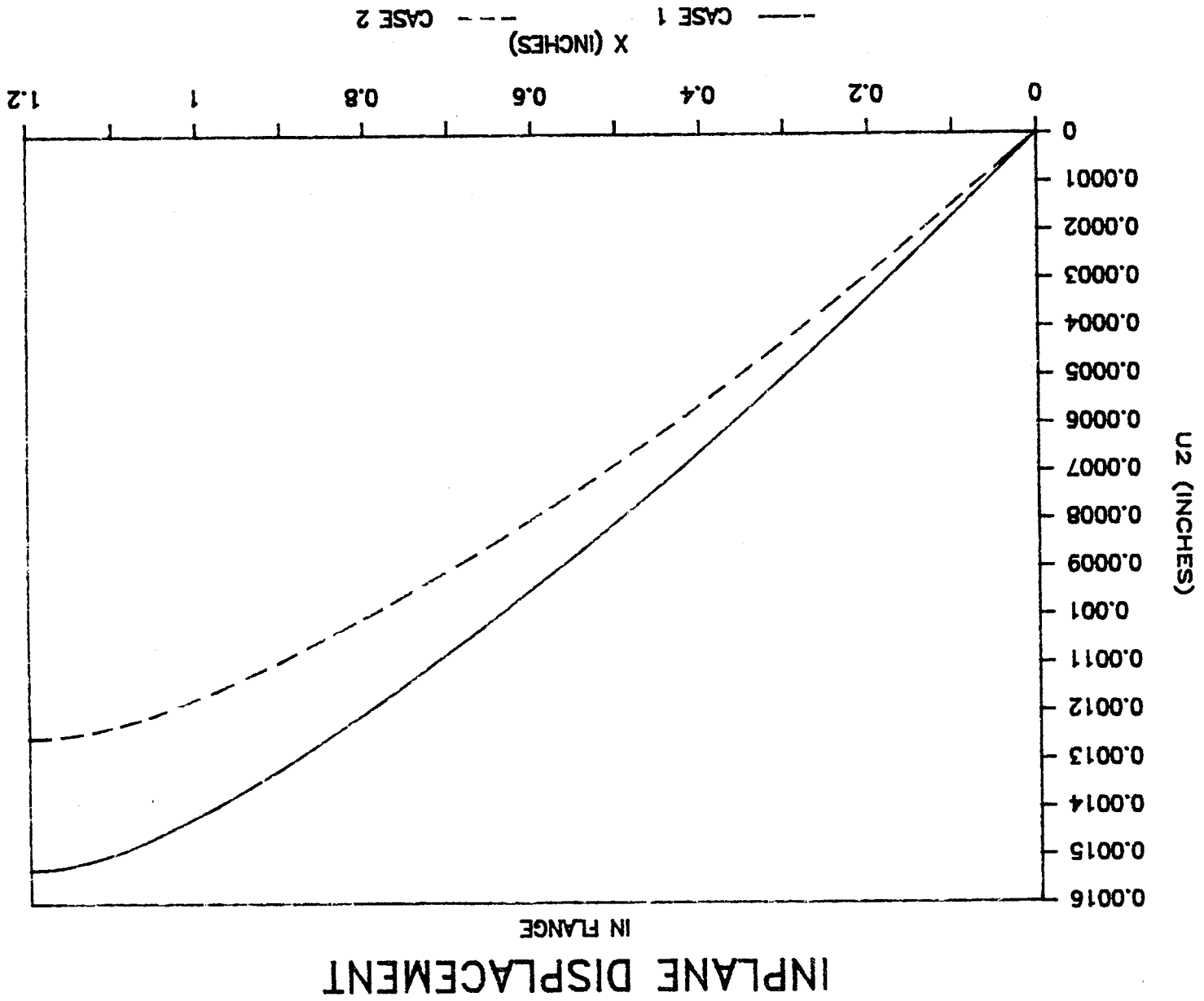


Figure 11. Inplane displacement of flange laminate.

5. SUMMARY

PSTRESS is an interactive computer program which calculates the deflections and stresses of a bonded skin/stringer combination with applied moments, inplane forces, and vertical shear forces. The results are within a few percent of those obtained by NASTRAN finite element analyses of equivalent structures. Using PSTRESS it is possible to evaluate the effects of loading and material parameters on the adhesive peel and shear stresses.

6. REFERENCES

1. Barkey, D. A., Madan, R. C., and Sutton, J. O. "Analytical Approach to Peel Stresses in Bonded Composite Stiffened Panels", 20th Midwestern Mechanics Conference on Composites, Purdue University, 1987, (also available as Douglas Paper No. 7907).

APPENDIX
PSTRESS Listing

**** P S T R E S S ****

**** PROGRAM TO COMPUTE RESPONSE OF BONDED ****
**** STRINGER/SKIN COMBINATION ****

**** ANALYSIS AND PROGRAM BY DEREK A. BARKEY ****
**** DOUGLAS AIRCRAFT COMPANY, JANUARY 27,1987 ****

SUBROUTINES NEEDED:

MATRIX-SOLVES SYSTEM OF ALGEBRAIC EQUATIONS GIVEN
THE MATRIX OF COEFFICIENTS

CMSXXX-ALLOWS THE EXECUTION OF CMS COMMANDS INSIDE
A FORTRAN PROGRAM

ROOT-SOLVES FOR ROOTS OF THIRD OR FOURTH ORDER POLYNOMIALS

OUTPUT SENT TO UNIT NUMBER 13, WHICH MUST BE DEFINED AS
"PSTRESS OUTPUT" BY A FILEDEF COMMAND. USER WILL BE PROMPTED
FOR THE PRINTING OF OUTPUT.

W1 AND W2 ARE DEFLECTIONS OF SKIN AND STRINGER FLANGE,
RESPECTIVELY. U1 AND U2 ARE AXIAL DISPLACEMENTS OF
SKIN AND STRINGER FLANGE, RESPECTIVELY. EP1 AND EP2
ARE STRAINS IN SKIN AND STRINGER FLANGE, RESPECTIVELY.
TAUC IS SHEAR STRESS IN ADHESIVE. SIGC IS PEEL STRESS IN
ADHESIVE.

FOR CASES WITH NO AXIAL FORCE:

$$W1=(C1*EXP(MU1*X)+C2*EXP(-MU1*X)+C3*EXP(MU2*X)+C4*EXP(-MU2*X)+C5*EXP(MU3*X)+C6*EXP(-MU3*X)+C7*X**3+C8*X**2+C9*X+C10)/D1$$

$$W2=(U21*(C1*EXP(MU1*X)+C2*EXP(-MU1*X))+U22*(C3*EXP(MU2*X)+C4*EXP(-MU2*X))+U23*(C5*EXP(MU3*X)+C6*EXP(-MU3*X))+A/B*(C7*X**3+C8*X**2+C9*X+C10))/D2$$

$$U1=(U51*(C1*EXP(MU1*X)-C2*EXP(-MU1*X))/MU1+U52*(C3*EXP(MU2*X)-C4*EXP(-MU2*X))/MU2+U53*(C5*EXP(MU3*X)-C6*EXP(-MU3*X))/MU3-3*(TT1+TT2*A/B)*C7*X**2/(F+G)+C11*X+C13)/(E1*T1)$$

$$U2=(-U51*(C1*EXP(MU1*X)-C2*EXP(-MU1*X))/MU1-U52*(C3*EXP(MU2*X)-C4*EXP(-MU2*X))/MU2-U53*(C5*EXP(MU3*X)-C6*EXP(-MU3*X))/MU3+3*(TT1+TT2*A/B)*C7*X**2/(F+G)+C12*X+C14)/(E2*T2)$$

$$EP1=(U51*(C1*EXP(MU1*X)+C2*EXP(-MU1*X))+U52*(C3*EXP(MU2*X)+C4*EXP(-MU2*X))+U53*(C5*EXP(MU3*X)+C6*EXP(-MU3*X))-6*(TT1+TT2*A/B)*C7*X/(F+G)+C11)/(E1+T1)$$

$$EP1=(-U51*(C1*EXP(MU1*X)+C2*EXP(-MU1*X))-U52*(C3*EXP(MU2*X)+C4*EXP(-MU2*X))-U53*(C5*EXP(MU3*X)+C6*EXP(-MU3*X))+6*(TT1+TT2*A/B)*C7*X/(F+G)+C12)/(E2+T2)$$

NOTE:

FOR CASES WITH AXIAL FORCE, SUBSTITUTE EXP(MU4*X) AND
EXP(-MU4*X) TERMS FOR X**3 AND X**2 TERMS.


```

C      SIG=EC*(W1-W2)/TC
C      TAU=GC*(2*(U1-U2)+D1*TT1*DW1/DX+D2*TT2*DW2/DX)/(2*TC)
C
C      NOTE:  C12=(2*(TT1+TT2*A/B)*C8+F*C11)/G
C              C14=(6*(TT1+TT2*A/B)*C7/(E*(F+G)))+(TT1+TT2*A/B)*C9+
C              F*C13)/G
C
C      CONSTANT          PROGRAM EQUIVALENT
C      A                AA
C      B                BB
C      E                EE
C      F                FF
C      G                GG
C      CI              E(I)
C      UIJ             U(I,J)
C      MUI             MU(I)
C      T1,T2           THICKNESS OF SKIN AND STRINGER FLANGE
C      D1,D2           RIGIDITY OF SKIN AND STRINGER FLANGE
C      E1,E2           MODULUS OF SKIN AND STRINGER FLANGE
C      EC,GC           EXTENSIONAL AND SHEAR MODULUS OF ADHESIVE
C      TC              THICKNESS OF ADHESIVE
C
C      CHARACTER ANSWER
C      CHARACTER*4 DNAME(2)
C      REAL*16 D1,D2,E1,E2,T1,T2,TC,GC,EC,L,P1,M1,F1,P2,M2,F2
C      REAL*16 AA,BB,CC,DD,EE,FF,GG,TT1,TT2,P,Q,R,S,RSIG1,RSIG2
C      REAL*16 RT,X,RU1,RU2,RW1,RW2,RSIG,RTAU,FLAG
C      COMPLEX*32 LAM,U,MU,A,B,D,E,F,G,W1,W2,U1,U2
C      COMPLEX*32 SIG,SIG1,SIG2,TAU
C      INTEGER J,N,I,M,COUNT
C      DIMENSION LAM(4),U(5,4),MU(4),A(12,12),B(12),E(14),X(20)
C      DIMENSION D(12,0:12),W1(20),W2(20),U1(20),U2(20),SIG(20)
C      DIMENSION SIG1(20),SIG2(20),TAU(20),RU1(20),RU2(20),RW1(20)
C      DIMENSION RW2(20),RSIG(20),RSIG1(20),RSIG2(20),RTAU(20)
C      CALL CMSXXX('CLRSCRN ',IRCODE)
C      WRITE(6,FMT='(//)')
C      WRITE(6,50)
C      WRITE(13,50)
50    FORMAT(T24,'P E E L   S T R E S S',/)
C      **** VALUES OF X FOR PLOTTING:  ALL THESE VALUES ****
C      **** WILL BE MULTIPLIED BY FLANGE WIDTH/1.2 ****
C      DATA (X(I),I=1,7)/0,.01875,.0375,.075,.1125,.15,.225/
C      DATA (X(I),I=8,14)/.3,.42,.54,.66,.78,.9,.975/
C      DATA (X(I),I=15,20)/1.05,1.0875,1.125,1.1625,1.18125,1.2/
C      **** INPUT PROPERTIES OF BEAMS AND ADHESIVE ****
C      WRITE(6,100)
C      READ*,E1,D1,T1
C      WRITE(13,150)E1,D1,T1
C      WRITE(6,200)
C      READ*,E2,D2,T2
C      WRITE(13,250)E2,D2,T2
C      WRITE(6,300)

```

```

READ*,EC,GC,TC
WRITE(13,350)EC,GC,TC
WRITE(6,400)
READ*,L
WRITE(13,450)L
100  FORMAT(' INPUT PROPERTIES OF SKIN:  E1, D1, T1')
150  FORMAT(' PROPERTIES OF SKIN:  ',1P1E10.3,0P1F10.1,1X,0P1F8.4)
200  FORMAT(' INPUT PROPERTIES OF FLANGE:  E2, D2, T2')
250  FORMAT(' PROPERTIES OF FLANGE: ',1P1E10.3,0P1F10.1,1X,0P1F8.4)
300  FORMAT(' INPUT ADHESIVE PROPERTIES:  EC, GC, TC')
350  FORMAT(' ADHESIVE PROPERTIES:  ',1P1E10.3,1X,1P1E10.3,0P1F8.4)
400  FORMAT(' INPUT FLANGE WIDTH:  L')
450  FORMAT(' FLANGE WIDTH:  ',F7.3)
500  FORMAT(' INPUT SHEAR, MOMENT, AND FORCE APPLIED TO SKIN')
550  FORMAT(/,' SHEAR, MOMENT, AND FORCE ON SKIN:  ',3F9.2)
600  FORMAT(' INPUT SHEAR, MOMENT, AND FORCE APPLIED TO FLANGE')
650  FORMAT(' SHEAR, MOMENT, AND FORCE ON FLANGE: ',3F9.2)
M=1
N=12
COUNT=0
C  **** DEFINE CONSTANTS ****
TT1=(T1+TC)/D1
TT2=(T2+TC)/D2
AA=EC/(TC*D1)
BB=EC/(TC*D2)
CC=GC*D1*TT1/(4.*TC)
DD=GC*D2*TT2/(4.*TC)
EE=GC/(2.*TC)
FF=2./(E1*T1)
GG=2./(E2*T2)
1500 COUNT=COUNT+1
U(1,4)=0
U(2,4)=0
U(3,4)=0
U(4,4)=0
U(5,4)=0
MU(4)=0
C  **** INPUT LOADING ON SKIN AND STRINGER FLANGE ****
WRITE(6,500)
READ*,P1,M1,F1
WRITE(13,800)COUNT
WRITE(13,550)P1,M1,F1
WRITE(6,600)
READ*,P2,M2,F2
WRITE(13,650)P2,M2,F2
800  FORMAT(/,20('-',),//,' CASE ',I2)
C  **** DEFINE COEFFICIENTS OF CHARACTERISTIC EQUATION ****
P=-((CC*TT1+F1/D1+DD*TT2+F2/D2+EE*(FF+GG))
Q=AA+BB+CC*TT1*F2/D2+DD*TT2*F1/D1+F1*F2/D1/D2+EE*(FF+GG)*
& (F1/D1+F2/D2)
R=-((AA+BB+F1*F2/D1/D2)*EE*(FF+GG)+(CC+DD)*(AA*TT2+BB*TT1)

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&   +BB*F1/D1+AA*F2/D2)
S=EE*(FF+GG)*(AA*F2/D2+BB*F1/D1)
IF (F1 .NE. 0 .OR. F2 .NE. 0) THEN
  FLAG=0
ELSE
  FLAG=1
ENDIF
C   **** SOLVE CHARACTERISTIC EQUATION ****
CALL ROOT(P,Q,R,S,LAM)
C   **** CALCULATE EIGENVECTORS ****
DO 1000 I=1,4-FLAG
  U(1,I)=1.
  U(2,I)=(AA*(CC+DD)-F1/D1*DD*LAM(I)+LAM(I)*LAM(I)*DD)/
&      (BB*(CC+DD)-F2/D2*CC*LAM(I)+LAM(I)*LAM(I)*CC)
  U(3,I)=LAM(I)
  U(4,I)=LAM(I)*U(2,I)
  U(5,I)=EE*LAM(I)*(TT1+TT2*U(2,I))/(LAM(I)-EE*(FF+GG))
  MU(I)=CQSQRT(LAM(I))
1000 CONTINUE
  IF (D1 .EQ. D2) THEN
    U(2,1)=-1.
    U(2,2)=-1.
  ENDIF
C   **** DEFINE MATRIX OF COEFFICIENTS FOR BOUNDARY CONDITIONS ****
C   **** NOTE: A*E=B ****
DO 1050 I=1,12
  DO 1025 J=1,12
    A(I,J)=(0,0)
1025 CONTINUE
1050 CONTINUE
DO 1100 I=1,4-FLAG
  A(5,2*I-1)=MU(I)
  A(5,2*I)=-MU(I)
  A(6,2*I-1)=MU(I)*U(2,I)
  A(6,2*I)=-MU(I)*U(2,I)
  A(3,2*I-1)=U(5,I)*CQEXP(MU(I)*L)
  A(3,2*I)=U(5,I)*CQEXP(-MU(I)*L)
  A(11,2*I-1)=-A(3,2*I-1)
  A(11,2*I)=-A(3,2*I)
  A(7,2*I-1)=LAM(I)*CQEXP(MU(I)*L)
  A(7,2*I)=LAM(I)*CQEXP(-MU(I)*L)
  A(8,2*I-1)=A(7,2*I-1)*U(2,I)
  A(8,2*I)=A(7,2*I)*U(2,I)
  A(4,2*I-1)=MU(I)*LAM(I)
  A(4,2*I)=-MU(I)*LAM(I)
  A(10,2*I-1)=U(2,I)
  A(10,2*I)=U(2,I)
&   A(1,2*I-1)=(CC*((TT1+F1/D1/CC+TT2*U(2,I))*MU(I)+(FF+GG)*
      U(5,I)/MU(I))-MU(I)*LAM(I))*CQEXP(MU(I)*L)
  A(1,2*I)=-A(1,2*I-1)*CQEXP(-2*MU(I)*L)
  A(2,2*I-1)=(DD*((TT1+(TT2+F2/D2/DD)*U(2,I))*MU(I)+(FF+GG)*

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&          U(5,I)/MU(I))-MU(I)*LAM(I)*U(2,I))*CQEXP(MU(I)*L)
A(2,2*I)=-A(2,2*I-1)*CQEXP(-2*MU(I)*L)
A(12,2*I-1)=U(5,I)/MU(I)
A(12,2*I)=-U(5,I)/MU(I)
A(9,2*I-1)=-U(5,I)/MU(I)
A(9,2*I)=U(5,I)/MU(I)
1100 CONTINUE
A(5,9)=1.
A(6,9)=AA/BB
A(3,7)=A(3,7)*(1-FLAG)-6.*(TT1+TT2*AA/BB)*L/(FF+GG)*FLAG
A(3,11)=1.
A(11,7)=A(11,7)*(1-FLAG)+6.*(TT1+TT2*AA/BB)*L/(FF+GG)*FLAG
A(11,8)=A(11,8)*(1-FLAG)+2.*(TT1+TT2*AA/BB)/GG*FLAG
A(11,11)=FF/GG
A(7,7)=A(7,7)*(1-FLAG)+6.*L*FLAG
A(7,8)=A(7,8)*(1-FLAG)+2.*FLAG
A(8,7)=A(8,7)*(1-FLAG)+6.*AA*L/BB*FLAG
A(8,8)=A(8,8)*(1-FLAG)+2.*AA/BB*FLAG
A(4,7)=A(4,7)*(1-FLAG)+6.*FLAG
A(10,10)=AA/BB
A(1,7)=A(1,7)*(1-FLAG)-6.*(1+CC*(TT1+TT2*AA/BB)/(EE*(FF+GG)))
&          *FLAG
A(1,9)=F1/D1
A(2,7)=A(2,7)*(1-FLAG)-6.*(AA/BB+DD*(TT1+TT2*AA/BB)/(EE*
&          (FF+GG)))*FLAG
A(2,9)=F2/D2*AA/BB
A(12,12)=1.
A(9,7)=A(9,7)*(1-FLAG)+6.*(TT1+TT2*AA/BB)/(EE*GG*(FF+GG))*FLAG
A(9,9)=(TT1+TT2*AA/BB)/GG
A(9,12)=FF/GG
C ***** BOUNDARY CONDITIONS *****
C
C ***** X=0  DW1/DX=0 *****
B(5)=0
C ***** X=0  DW2/DX=0 *****
B(6)=0
C ***** X=L  E1*T1*DU1/DX=F1 *****
B(3)=F1
C ***** X=L  E2*T2*DU2/DX=F2 *****
B(11)=F2
C ***** X=L  D1*D**2W1/DX**2=M1 *****
B(7)=M1
C ***** X=L  D2*D**2W2/DX**2=M2 *****
B(8)=M2
C ***** X=0  D**3W1/DX**3=0 *****
B(4)=0
C ***** X=0  W2=0 *****
B(10)=0
C ***** X=L  SHEAR CONDITION *****
B(1)=P1
C ***** X=L  SHEAR CONDITION *****
B(2)=P2

```

```

C      **** X=0  U1=0  ****
      B(12)=0
C      **** X=0  U2=0  ****
      B(9)=0
C
C      **** SOLVE FOR E ****
      CALL MATRIX(A,B,E,M,N)
C      **** PLUG E INTO A*E=B, TO CONFIRM E ****
      DO 2250 I=1,12
          DO 2230 J=1,12
              D(I,J)=A(I,J)*E(J)
2230      CONTINUE
2250      CONTINUE
      E(13)=E(12)
C      **** SOLVE FOR C12 AND C14 ****
      E(12)=(2.*(TT1+TT2*AA/BB)*E(8)*FLAG+FF*E(11))/GG
      E(14)=(6.*(TT1+TT2*AA/BB)*E(7)*FLAG/(EE*(FF+GG))
&      +(TT1+TT2*AA/BB)*E(9)+FF*E(13))/GG
C      **** PRINT OUTPUT TO FILES AND SCREEN (IF DESIRED) ****
      CALL CMSXXX('CLRSCRN ',IRCODE)
      WRITE(13,2100) (I,LAM(I),I,MU(I),I=1,4)
      WRITE(13,2200) ((I,J,U(I,J),J=1,4),I=1,5)
      WRITE (13,2300) (I,E(I), I=1,14)
      ANSWER='N'
      WRITE(6,FMT='(///, ' DO YOU WANT TO SEE EIGENVALUES, ',
& ' EIGENVECTORS, AND COEFFICIENTS(Y/N)?',/,
& ' "NO" IS DEFAULT.'))
      READ(5,5300,END=2095)ANSWER
2095      CALL CMSXXX('CLRSCRN ',IRCODE)
      IF (ANSWER .NE. 'Y') GOTO 2805
      WRITE(6,2100) (I,LAM(I),I,MU(I),I=1,4)
      WRITE(6,2200) ((I,J,U(I,J),J=1,4),I=1,5)
      WRITE (6,2300) (I,E(I), I=1,14)
2100      FORMAT(/, ' EIGENVALUES AND THEIR SQUARE ROOTS:',
&      (4(/, ' LAM(' ,S,I1, ' )',1X,1P1E15.7,SP,1P1E15.7,'i',
&      /, ' MU(' ,S,I1, ' )',2X,1P1E15.7,SP,1P1E15.7,'i',/)))
2200      FORMAT(' EIGENVECTORS:',5(/,
&      4(' U(' ,S,I1, ' , ' ,S,I1, ' )',1X,1P1E15.7,SP,1P1E15.7,'i',/)))
2300      FORMAT(' COEFFICIENTS ARE:',/,
&      14(' C(' ,S,I2, ' )',2X,1P1E15.7,SP,1P1E15.7,'i',/))
C      **** SUM ROWS OF D AND THEN COMPARE WITH B ****
2805      DO 2830 I=1,12
          D(I,0)=B(I)
          DO 2820 J=1,12
              D(I,0)=D(I,0)-D(I,J)
2820      CONTINUE
2830      CONTINUE
      DO 2900 I=1,12
          IF (CQABS(D(I,0)) .GT. 1.Q-20) THEN
              D(I,0)=-D(I,0)+B(I)
              WRITE(6,3000)I,B(I),D(I,0)

```

```

        WRITE(13,3000) I,B(I),D(I,0)
    ENDIF
2900  CONTINUE
3000  FORMAT(' BOUNDARY CONDITION ',S,I2,
& ' NOT SATISFIED BY COEFFICIENTS.',/, ' CONDITION EQUALS:',
& 1P1E15.7,SP,1P1E15.7,'i',/, ' SHOULD EQUAL: ',S,
& 1P1E15.7,SP,1P1E15.7,'i')
C     **** CALCULATE DISPLACEMENTS AND STRESSES FOR VARIOUS ****
C     **** VALUES OF X, AND PRINT THEM ****
    DO 3100 I=1,20
        X(I)=X(I)*L/1.2
3100  CONTINUE
    DO 4000 I=1,20
        W1(I)=0
        W2(I)=0
        SIG1(I)=0
        SIG2(I)=0
        U1(I)=0
        U2(I)=0
        TAU(I)=0
        DO 3200 J=1,4-FLAG
            F=E(2*J-1)*CQEXP(MU(J)*X(I))
            G=E(2*J)*CQEXP(-MU(J)*X(I))
            W1(I)=W1(I)+F+G
            W2(I)=W2(I)+(F+G)*U(2,J)
            SIG1(I)=SIG1(I)+(F+G)*U(5,J)
            SIG2(I)=SIG2(I)-(F+G)*U(5,J)
            U1(I)=U1(I)+(F-G)*U(5,J)/MU(J)
            U2(I)=U2(I)-(F-G)*U(5,J)/MU(J)
            TAU(I)=TAU(I)+EE*((FF+GG)*U(5,J)/MU(J)
&                +(TT1+TT2*AA/BB)*MU(J))*(F-G)
3200  CONTINUE
        RW1(I)=REAL(W1(I)+E(7)*FLAG*X(I)**3.+E(8)*FLAG*X(I)*X(I)
&                +E(9)*X(I)+E(10))/D1
&
        RW2(I)=REAL(W2(I)+AA/BB*(E(7)*FLAG*X(I)**3.+E(8)*FLAG
&                *X(I)*X(I)+E(9)*X(I)+E(10)))/D2
&
        RSIG1(I)=REAL(SIG1(I)-6.*(TT1+TT2*AA/BB)*E(7)*FLAG*X(I)/
&                (FF+GG)+E(11))/T1
&
        RSIG2(I)=REAL(SIG2(I)+6.*(TT1+TT2*AA/BB)*E(7)*FLAG*X(I)/
&                (FF+GG)+E(12))/T2
&
        RU1(I)=REAL(U1(I)-3.*(TT1+TT2*AA/BB)*E(7)*FLAG*X(I)*X(I)/
&                (FF+GG)+E(11)*X(I)+E(13))/(E1*T1)
&
        RU2(I)=REAL(U2(I)+3.*(TT1+TT2*AA/BB)*E(7)*FLAG*X(I)*X(I)/
&                (FF+GG)+E(12)*X(I)+E(14))/(E2*T2)
&
        RSIG(I)=(RW1(I)-RW2(I))*EC/TC
        RTAU(I)=REAL(TAU(I)+EE*(FF*(-3.*(TT1+TT2*AA/BB)*E(7)*FLAG
&                *X(I)*X(I)/(FF+GG)+E(11)*X(I)+E(13))-GG*(3.*
&                (TT1+TT2*AA/BB)*E(7)*FLAG*X(I)*X(I)/(FF+GG)+E(12)
&                *X(I)+E(14))+(TT1+TT2*AA/BB)*(3.*E(7)*FLAG*X(I)
&                *X(I)+2.*E(8)*FLAG*X(I)+E(9))))
4000  CONTINUE

```

```

WRITE(6,4200)
WRITE(13,4200)
WRITE(6,4300)
WRITE(13,4300)
4200 FORMAT(/,34H                                DEFLECTIONS)
4300 FORMAT(51H          X          W1          W2          U1          U2)
WRITE(6,4500)(X(I),RW1(I),RW2(I),RU1(I),RU2(I), I=1,20)
WRITE(13,4500)(X(I),RW1(I),RW2(I),RU1(I),RU2(I), I=1,20)
4500 FORMAT(20(/,0P1F11.4,4(1P1E11.3)))
WRITE(6,4700)
WRITE(13,4700)
WRITE(6,4800)
WRITE(13,4800)
4700 FORMAT(/,32H                                STRESSES)
4800 FORMAT(52H          X          SIGC          TAUC          SIG1          SIG2)
WRITE(6,5000)(X(I),RSIG(I),RTAU(I),RSIG1(I),RSIG2(I),
&          I=1,20)
WRITE(13,5000)(X(I),RSIG(I),RTAU(I),RSIG1(I),RSIG2(I),
&          I=1,20)
5000 FORMAT(20(/,0P1F11.4,4(1P1E11.3)))
5100 WRITE(6,5200)
REWIND(5)
READ(5,5300,END=6000)ANSWER
IF (ANSWER .EQ. 'Y') THEN
    CALL CMSXXX('CLRSCRN ',IRCODE)
    GOTO 1500
ENDIF
5200 FORMAT(/,' ANOTHER RUN WITH DIFFERENT LOADS(Y/N)?')
5300 FORMAT(A1)
6000 WRITE(6,6200)
REWIND(5)
READ(5,6300,END=7000)ANSWER
IF (ANSWER .NE. 'Y') GOTO 7000
WRITE(6,6400)
REWIND(5)
READ(5,6500,END=6100)DNAME
CALL CMSXXX('EXEC      ','DSPR      ','PSTRESS      ','OUTPUT      ',
&          'A          ','(          ','DNAME,IRA)')
CALL CMSXXX('CP        ','SLEEP      ','IRA)')
GOTO 7000
6100 CALL CMSXXX('EXEC      ','DSPR      ','PSTRESS      ','OUTPUT      ',IRCC)
CALL CMSXXX('CP        ','SLEEP      ','IRA)')
6200 FORMAT(/,' SEND RESULTS TO PRINTER(Y/N)?')
6300 FORMAT(A1)
6400 FORMAT(/,' ENTER PRINTER DESTINATION:  <ENTER> FOR DEFAULT')
6500 FORMAT(2A4)
7000 CALL CMSXXX('CLRSCRN ',IRCODE)
STOP
END
SUBROUTINE ROOT(P,Q,R,S,LAM)
C      **** SOLVES FOR ROOTS OF CHARACTERISTIC EQUATION (LAMBDA) ****

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REAL*16 A,B,C,AP,BP,RT,P,Q,R,S
COMPLEX*32 L,M,N,AS,BS,LAM
DIMENSION LAM(4)
IF (S .EQ. 0) THEN
  A=P
  B=Q
  C=R
ELSE
  A=(Q-3.*P*P/8.)/2
  B=((Q-3.*P*P/8.)*(Q-3.*P*P/8.)-4.*(-3.*P*P*P*P/256.
&   +P*P*Q/16-R*P/4.+S))/16.
  C=-(P*P*P/8.-P*Q/2.+R)*(P*P*P/8.-P*Q/2.+R)/64.
ENDIF
AP=B-A*A/3.
BP=(2.*A*A*A-9.*A*B+27.*C)/27.
AS=-BP/2.+CQSQRT(QCMPLX(BP*BP/4.+AP*AP*AP/27.))
AS=CQABS(AS)**(1./3.)*CQABS(AS)/AS
BS=-BP/2.-CQSQRT(QCMPLX(BP*BP/4.+AP*AP*AP/27.))
BS=CQABS(BS)**(1./3.)*CQABS(BS)/BS
RT=3.
L=-.5*(AS+BS)+(0,1.)*QSQRT(RT)/2.*(AS-BS)-A/3.
M=-.5*(AS+BS)-(0,1.)*QSQRT(RT)/2.*(AS-BS)-A/3.
N=AS+BS-A/3.
IF (S .EQ. 0) THEN
  LAM(1)=L
  LAM(2)=M
  LAM(3)=N
  LAM(4)=0
ELSE
  L=CQSQRT(L)
  M=CQSQRT(M)
  N=CQSQRT(N)
  BP=P*P*P/8.-P*Q/2.+R
  LAM(3)=QABS(BP)/BP*(-L-N-M)-P/4.
  LAM(2)=QABS(BP)/BP*(L-M+N)-P/4.
  LAM(1)=QABS(BP)/BP*(-L+M+N)-P/4.
  LAM(4)=QABS(BP)/BP*(L+M-N)-P/4.
ENDIF
RETURN
END
SUBROUTINE MATRIX(D,C,B,M,N)
PURPOSE
  THIS SUBROUTINE SOLVES A SYSTEM OF LINEAR ALGEBRAIC
  EQUATIONS BY TRIPLE FACTORIZATION WITH PARITAL PIVOTING
  AND BACK SUBSTITUTION.

TRIPLE MATRIX FACTORIZATION
*   DECOMPOSE A COMPLEX MATRIX INTO ITS COMPONENT
*   MATRICES IN THE MANNER A=LDU, WHEREIN
*   A IS THE GIVEN MATRIX
*   L IS A LOWER TRIANGULAR MATRIX OF UNIT DIAGONAL

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C      *           D IS A DIAGONAL MATRIX
C      *           U IS AN UPPER TRIANGULAR MATRIX OF UNIT DIAGONAL
C      *
C      * DESCRIPTION
C      * THE ORDER OF DECOMPOSITION IS IN THE SEQUENCE OF
C      * COLUMNS. THE LARGEST ABSOLUTE ELEMENT BELONGING
C      * TO A ROW AS YET NOT DECOMPOSED IS USED AS PIVOT.
C      * THE COMPONENT MATRICES SUPERIMPOSE THE ORIGINAL.
C
C      * ARG.  TYPE I/O/S  DIMS.  DEFINITION
C      * A     COMPLEX*32  (N,N)  INPUT AND DECOMPOSED MATRICES
C      * B     COMPLEX*32  (N,M)  B ON INPUT, X ON OUTPUT
C      * KOL   INTEGER     N      ROW ORDER OF DECOMPOSITION
C      * S     COMPLEX*32  N      SCRATCH ARRAY
C      * N     INTEGER     N      ORDER OF THE SYSTEM
C      * M     INTEGER     N      NUMBER OF COLUMNS
C
C      * CODED BY L. CHAHINIAN, 7/16/80
C      * MODIFIED BY D. BARKEY FOR USE IN PEEL STRESS COMPUTATION
C      * PROGRAM "PSTRESS", 2/20/87

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DIMENSION D(N,N),C(N,M),B(N,M)
DIMENSION KOL(20),S(20),A(20,20)
REAL*16 R
COMPLEX*32 A,Q,S,B,C,D
EQUIVALENCE (Q,R)
DO 200 I=1,N
  DO 100 J=1,N
    A(I,J)=D(I,J)
100  CONTINUE
  DO 150 J=1,M
    B(I,J)=C(I,J)
150  CONTINUE
200  CONTINUE
DO 1000 I=1,N
1000 KOL(I)= I
    NM1 = N-1
    DO 4000 K=1,NM1
C      **** DETERMINE LARGEST ABSOLUTE ELEMENT IN THIS COLUMN ****
        KP1 = K+1
        R = CQABS(A(K,K))
        L = K
        DO 1200 I=K,N
          IF(CQABS(A(I,K)).LE.R) GO TO 1200
          R = CQABS(A(I,K))
          L = I
1200 CONTINUE
        S(K)= A(L,K)
        IF(L.EQ.K)GO TO 1241
        J = KOL(K)
        KOL(K)= KOL(L)
        KOL(L)= J

```


```

C      **** INTERCHANGE ROWS    K AND L ****
      DO 1220 I=1,N
      Q      = A(K,I)
      A(K,I)= A(L,I)
1220  A(L,I)= Q
1241  Q      = A(K,K)
      DO 2000 I=KP1,N
      A(I,K)= A(I,K)/Q
      DO 2000 J=KP1,N
      IF (CQABS(A(K,J)) .LT. 1.Q-35) A(K,J)=(0.,0.)
2000  A(I,J)= A(I,J)-A(I,K)*A(K,J)
      DO 4000 J=KP1,N
4000  A(K,J)= A(K,J)/Q
C
C      **** SOLVE FOR UNKNOWNNS, GIVEN TRIPLE FACTORIZATION ****
C
C      *      SOLVE FOR X IN THE MATRIX EXPRESSION AX=B
C      *      WHERE THE COLUMNS OF B ARE GIVEN, AND
C      *      WHERE A IS GIVEN IN TERMS OF ITS COMPONENT
C      *      MATRICES L, D, AND U, SUCH THAT
C      *      A = L D U
C      *      L IS A LOWER TRIANGULAR MATRIX OF UNIT DIAGONAL
C      *      D IS A DIAGONAL MATRIX
C      *      U IS AN UPPER TRIANGULAR MATRIX OF UNIT DIAGONAL
C      *      DESCRIPTION
C      *      THE COMPONENT MATRICES ARE BROUGHT THROUGH ARRAY A.
C      *      THEIR ROWS HAVE BEEN INTERCHANGED PER INFORMATION
C      *      ARRAY KOL. FOR EACH REQUIRED COLUMN, THE CONTENTS
C      *      ARE FIRST TRANSFERRED INTO STORAGE ARRAY S, AND
C      *      REARRANGED BACK INTO THE ORIGINAL COLUMN PER KOL.
C      *      A FORWARD PASS OF THAT COLUMN ON MATRIX L YIELDS
C      *      Y, WHERE LY=B. THIS IS FOLLOWED WITH THE
C      *      COMPUTATION OF Z VIA DZ=Y, AND AT LAST UX=Z
C      *      PROVIDES X THROUGH A REVERSE PASS ON U. THE
C      *      CONTENTS OF ARRAY B HAVE THUS BEEN REPLACED
C      *      WITH X.
C
C      FOR EACH COLUMN OF B
      DO 10000 J=1,M
C      INTERCHANGE ROWS OF B PER KOL
      DO 6100 I=1,N
6100  S(I)= B(I,J)
      DO 6200 I=1,N
      K      = KOL(I)
6200  B(I,J)= S(K)
C      **** FOR EACH COLUMN OF B ****
C      **** SOLVE LY=B, FOR Y ****
      IM1    = 1
      DO 8000 I=2,N
      DO 7000 L=1,IM1
      B(I,J)=B(I,J)-A(I,L)*B(L,J)

```

```
7000 CONTINUE
8000 IM1 = I
C **** SOLVE DZ=Y, FOR Z ****
DO 9000 I=1,N
9000 B(I,J)= B(I,J)/A(I,I)
C **** SOLVE UX=Z, FOR X ****
I = N
GO TO 9700
9100 DO 9300 L=IP1,N
9300 B(I,J)= B(I,J)-A(I,L)*B(L,J)
9700 IP1 = I
I = I-1
IF(I.GT.0)GO TO 9100
10000 CONTINUE
RETURN
END
```

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16. Abstract The report describes the use of the interactive FORTRAN computer program PSTRESS, which computes a closed form solution for two bonded plates subjected to applied moments, vertical shears, and inplane forces. The program calculates inplane stresses in the plates, deflections of the plates, and peel and shear stresses in the adhesive. The document briefly outlines the analytical method used by PSTRESS, describes input and output of the program, and presents a sample analysis. The results of this sample analysis are shown to be within a few percent of results obtained using a NASTRAN finite element analysis. An appendix containing a listing of PSTRESS is included.					
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