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## -ModatAnalysis of UH-60A

Instrumented Rotor Blades


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# Modal Analysis of UH-60A Instrumented Rotor Blades 

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## SUMMARY

The dynamic characteristics of instrumented and production UH-60A Black Hawk main rotor blades were measured, and the results were validated with NASTRAN finite element models. The blades tested included pressure and strain-gage instrumented blades, which are part of NASA's Airloads Flight Research Phase of the Modern Technology Rotor Program. The dynamic similarity of the blades was required for accurate data collection in this program. Therefore, a nonrotating blade modal analysis was performed on the first 10 free-free modes to measure blade similarities. The results showed small differences between the modal frequencies of instrumented and production blades and a close correlation with the NASTRAN models. This type of modal testing and analysis is recommended as a standard procedure for future instrumented blade flight testing.

## INTRODUCTION

This report documents the dynamic similarities of the UH-60A Black Hawk blades used in the Airloads Flight Research Phase of the Modern Technology Rotor Program (ref. 1). Two of the four blades on the research aircraft are highly instrumented, one with pressure transducers and the other with strain gages (fig. 1). Although the blades were built to form a matched set, it was important to verify that the dynamic properties of the instrumented blades were the same as production blades.

The blade similarities were determined by performing a shake test. In this test, the nonrotating natural frequencies below 100 Hz were measured for each blade. Also, full mode shapes were measured for the two instrumented blades and one production blade. Three additional production blades were tested to measure their typical frequency variation. Furthermore, NASTRAN finite element models were created to model the vibration characteristics of the standard and instrumented blades (ref. 2). Then, the results from the shake test were used to validate the structural dynamic models of the blades.

It was important to verify that the instrumented blades had the same dynamic characteristics as production blades, because rotorcraft analysis programs generally assume that identical blades are used. The airloads data collected from the instrumented blades will be used to validate rotorcraft computer codes for rotors operating at high speeds and to fine tune the codes for better correlation with flight data. Therefore, the goal of the present study was to validate that assumption and to ensure the usefulness of the airloads data collected.

The testing and analysis of the blade dynamics are part of an experiment supporting the airloads program. Ames Research Center, in cooperation with the U.S. Army (AVSCOM), is conducting the research to develop a greater understanding of modem technology rotor systems. The program will acquire the most comprehensive rotor airloads study to date to provide the prime element of an extensive rotorcraft aerodynamic and dynamic data base.

The authors thank Chris Chen and Frank Pichay for their invaluable assistance in setting up the test. We also appreciate John Madden's insight on modal analysis and Anna Hood's help with the modal data.

## NASTRAN FINITE ELEMENT MODELS

The NASTRAN finite element models were created by NASA to reflect the structural characteristics of the production and instrumented blades. The strain-gage blade is instrumented with 68 strain gages and 20 accelerometers embedded under the skin of the blade. The pressure-instrumented blade has even more instrumentation with 242 pressure transducers, 50 temperature gages, and 5 strain gages. The location of the pressure transducers is shown in figure 2 for the top and botom blade surfaces. Figure 3 is a photograph of the pressure blade tip showing the pressure transducers embedded under the skin.

For each blade, the major load-carrying members are unchanged, including the titanium spar, blade attachment cuff fitting, and the root-end graphite laminates (ref. 3). Modifications were made, however, to the fiberglass skin, the leading-edge sheath, and the counterweights. Sikorsky Aircraft added the instrumentation in a way that minimizes changes in the mass and center of gravity of the blades.

The three production blades were also slightly modified. They were painted white on top to match the instrumented blades and a set of wires was added along the bottom to accommodate tip accelerometers. The straingage blade and three production blades were statically and dynamically balanced on a whirl stand to the pressure blade to form a five-blade set. A fourth production blade was not part of the balanced set, but it was used as an additional comparison to the instrumented blades.

## Model of the Production UH-60A Rotor Blade

The UH-60A has modem technology blades with $-18^{\circ}$ of equivalent linear twist (ref. 4). Furthermore, each blade has two different airfoils (SR1095 and 1095R8),
and the tip is swept by $20^{\circ}$. The primary structural elements are a titanium spar, a nomex honeycomb core, fiberglass skin, and an aluminum tip cap. The length of the blade is 24.15 ft with a rotor diameter of 53.67 ft . The blade has a $1.74-\mathrm{ft}$ chord and weighs 211.7 lb .

A NASTRAN model of a production UH-60A blade was created and used as a baseline to predict blade dynamic behavior. The finite element model was constructed as an equivalent beam model such as those used in rotor codes. The structural properties used to create the model were documented by Sikorsky Aircraft (ref. 5). These parameters were given as a function of radial location out from the hub. Therefore, it was possible to model the blade as a single row of beam elements. The structural characteristics included in each element were: twist; center of gravity (c.g.); elastic axis position; mass per unit length; mass c.g.; torsional moment of inertia about the c.g.; and flapwise, chordwise, torsional, and axial stiffness.

The blade was modeled in NASTRAN as a linear isotropic beam with bending, extension, and torsion in two perpendicular planes (fig. 4). The NASTRAN element used was the CBEAM. This element is like the more common CBAR element, but it has added capabilities such as specifying the mass c.g. offset from the elastic axis and tapered properties along the length of the element. The model also included the elastic axis (which is not straight) and the tip sweep angle. These features allowed for accurate modeling of the discontinuous blade properties. The NASTRAN code for the production blade is given in appendix $A$.

## Instrumented-Blade Models

A model was also created to analyze the frequencies and mode shapes of the pressure-instrumented blade. Sikorsky Aircraft supplied the structural differences of the blade in terms of four parameters: blade spanwise mass distribution, mass c.g. position from the elastic axis, and the flapwise and chordwise stiffnesses. The strain-gage blade differed only in terms of the c.g. and mass with the largest deviation at $96 \%$ of the blade radius, giving a mass decrease and a corresponding c.g. farther aft. Since the strain-gage blade had minimal alteration, the productionblade model was sufficient for its analysis.

The model of the pressure-instrumented blade was created by modifying the production-blade model. The mass was larger along most of the blade length and the c.g. was forward that of production blades. However, at the tip, the mass was decreased and the c.g. shifted aft. The flapwise and chordwise stiffnesses were lower than
that of a production blade from 63 to $93 \%$ radius. Figure 5 shows the chordwise stiffness along the length of a production blade and the lower stiffness of the pressureinstrumented blade. This reduced stiffness and the decreased tip mass was a result of the removal of the nickel abrasion strip along the outboard leading edge (ref. 3). The higher mass was a result of the many wires needed for the instrumentation despite the mass removed as compensation. Appendix B contains the NASTRAN model of the pressure-instrumented blade.

## Modal Analysis

The finite element models were analyzed using the NASTRAN normal modes solution which calculated the natural frequencies and mode shapes of a single blade. First, free-free boundary conditions were used to correlate with the shake test. Then, the boundary conditions of the shaker and support system were modeled to study any stiffening effects they had on the blade motion. The bungee cords were modeled as four CELAS2 spring elements. The shaker consisted primarily of a CELAS2 element to model the diaphragm and a CONM2 for the moving weight element in the shaker. A CBAR element modeled the stainless steel stinger. Static tests were performed to measure the stiffnesses of the bungee cords and stinger. The shake test equipment and set-up are described in the next section.

A further analysis calculated any increase in natural frequency when a uniform gravity field was applied. This was done using two solution sequences. The NASTRAN Solution 64 (Geometric Nonlinear Analysis) was first used to apply a static gravity load of 1 g on the blade (ref. 6). This solution calculated any incremental stiffness due to the gravity load and stored the information in a data base. A subsequent run with Solution 63 (Normal Modes, Data Base) used this information to calculate the natural frequencies. The final correlation was made with the boundary conditions added to the model and the effects of gravity included.

## SHAKE TEST

The shake test of the UH-60A blades measured the free-free frequencies and mode shapes of the blades. This approach was selected instead of a clamped or pinned blade because of the ease of conducting the experiment in the free-free configuration. The objective of the test was to compare the blade's vibration characteristics. Therefore, measuring their natural frequencies was the most straightforward method. A comparison of the natural modes gave a direct indication of the blades' dynamic similarity in flight.

## Set-up

The test was performed at Ames Research Center in the Engineering Test Laboratory. Four bungee cords suspended the blades from a ceiling support beam (fig. 6). The cords extended from the blade retention pins at the root to a bracket atuached to the ceiling (fig. 7). Each loop had a stiffness of $50 \mathrm{lb} / \mathrm{ft}$. The rope and pulley apparatus shown in the photograph was a safety support and did not suspend the blade during the test.

A true free-free set-up would have no constraints on the blade during the test. However, the best approximation to this was to hang the blade from bungee cords whose stiffness was lower than that of the blade.

Also shown in figure 7 is the electrodynamic shaker. A $90^{\circ}$ angle plate, which was clamped to a vertical steel I-beam near the blade, supported the shaker. The weight of the moving mass element in the shaker was 0.25 lb . The effective spring stiffness of the shaker diaphragm was measured to be approximately $353 \mathrm{lb} / \mathrm{ft}$.

A force link or stinger attached the shaker to the blade. The stinger was made of stainless steel tubing 4 in . long and $1 / 16 \mathrm{~h}$ in. in diameter. It had a low flexural stiffness of $0.13 \mathrm{lb} / \mathrm{ft}$ and a high axial compression stiffness to minimize bending moments input to the blade during excitation. A bracket attached the stinger to the blade at the driving point. The root driving point was chosen based on the results from a past shake test performed on a UH60A blade (ref. 7). Also, the NASTRAN model indicated that the root was not a node point for the blade.

The shaker shown in figure 7 was oriented to excite the chordwise modes. However, the blade was rotated $90^{\circ}$ to excite the flapwise and torsional modes. Due to the large twist of UH-60A blades, a flapwise force from the shaker easily excited the torsional modes.

Figure 8 shows the schematic of the test set-up. Signals for driving the electrodynamic shaker originated from the built-in signal generator of the Hewlett Packard 3562A dynamic signal analyzer. These signals were then amplified and sent to the shaker. The analyzer measured the applied load at the driving point as the system input. A movable piezoelectric accelerometer measured the response of the blade at various measurement points. The accelerometer signals were sent through an amplifier to the analyzer as the system output.

The analyzer calculated a frequency response function (FRF) by dividing the Fourier transform of the output by the transform of the input. The FRF contained the
magnitude and phase information at each accelerometer location. This information was then stored on magnetic disk for further analysis by a modal software package (SMS Modal 3.0) (ref. 8).

This set-up was virtually identical to the set-up of the earlier UH-60A blade shake test (ref. 7). Therefore, it was assumed that data contamination from unmeasured shaker input and reciprocity characteristics previously measured held for this tes.

## Procedures

Accelerometer measurements were taken at 39 locations on the leading and trailing edges of the blade at approximately 16 in . intervals along the radius (fig. 9 , table 1). Accelerometer wax and mounting blocks aligned the accelerometer when necessary in the chordwise and flapwise axes. The frequency range of interest for the blade was from 0 to 100 Hz . During preliminary impact testing and using the results of the previous test, 10 modes were identified in this range. These included six flapwise, two chordwise, and two torsional modes. At each location all 10 modes were excited and measured before the accelerometer was moved to the next point.

A linear sine sweep excited each mode. This allowed fast measurements and a lower input force level compared to random input. A $1-\mathrm{Hz}$ bandwidth was centered around each mode. One sweep was made for each of the 10 modes. A very low bandwidth was acceptable because of low damping and well separated modes in the blade.

The analyzer controlled the force level to excite the blade. To determine the input force level to be used, FRFs were measured while the input force level was varied. In a linear system, an FRF has a constant value with varying input force amplitude. The lowest force level that excited the blade in the linear range was used. A force level of 0.5 lb was used for all of the modes except the first flapping mode which had a $0.4-\mathrm{lb}$ force.

## RESULTS

## Test Results of Instrumented and Production Blades

Figure 10 shows the first flapwise mode of production blade 1 and the instrumented blades measured in the shake test. The lines indicate the amplitude and phase at each accelerometer location calculated by the modal analysis software (ref. 8). The first flapwise frequency of the strain-gage blade is 4.78 Hz , which is $1 \%$ less than the production blade. The pressure-instrumented blade mode is 4.69 Hz , which is $3 \%$ less than the production blade.

Although the instrumented blades have slightly lower frequencies, their mode shapes have essentially the same characteristic motion.

A comparison of all 10 modes of the instrumented and production blades is shown in figure 11 and in table 2. The production blade results in figure 11 represent an average of the four standard blades tested. Modal residues are listed in table 3. These modes have a unit modal mass normalization. The phase at each measurement point should be either 0 or $180^{\circ}$. The amount of variation indicates the degree of coherence. A comparison of the mode shapes shows that the blade dynamics are nearly identical.

The trend of slightly lower instrumented blade frequencies is consistently seen for all 10 modes in figure 11. The strain-gage blade frequencies were within $0.7 \%$ of the production blade test average. The pressure-instrumented blade frequencies were systematically lower than those of a production blade by 2.0 to $3.7 \%$. This is a result of the increased weight and decreased stiffness of the blades due to the added instrumentation. Weights of the blades are shown in table 2.

The modes of the instrumented blades were compared with proprietary data provided by Sikorsky Aircraft. Sikorsky measured the first chordwise frequency of nearly 100 production blades. The first chordwise mode of the strain-gage blade fell within the typical variation of production blades. The pressure-instrumented blade was below the lowest frequencies Sikorsky measured.

The modal damping was small-between 0.1 and $2.4 \%$ critical damping. The first flapwise mode had the highest damping as a result of large contributions from the shaker and bungee attachments. Data from the previous shake test pointed to this conclusion because it had a different bungee attachment and lower damping (ref. 7). Also, the damping measured for the first mode was sensitive to stinger position. As the bungee cords stretched over the test period, the shaker height was adjusted to maintain a level stinger position.

The rigid body modes were measured with the shaker detached, letting the blade swing freely from the bungee cords. They were 0.2 Hz for both the chordwise and flapwise directions.

## Frequency Variation of Production BladesNASTRAN Correlation

Results from the shake test analyses of the four production blades showed that there is a large variation among their modal frequencies up to $2.4 \%$. Their natural
frequencies are given in figure 12 and in table 2. The standard deviation varied from 0.025 to 0.93 Hz . The first blade varied most with frequencies between 0.8 and $2.4 \%$ higher than the other three blades. Blades 2,3, and 4 were more closely matched.

The first blade was retested at the end of the project measuring the same frequencies. This retest verified the repeatability of the experiment. The production blade frequencies were also compared with the frequencies provided by Sikorsky Aircraft. All four blades fell within the typical frequency range. Therefore, the differences between the production blades were determined to be a result of production tolerance.

The NASTRAN model correlated well with the shake test results (fig. 12). The calculated mode shapes were within the $2.4 \%$ variation of the measured frequencies. With free-free boundary conditions applied, the NASTRAN modes were between 0.1 and $2.4 \%$ of the shake test. The exception was the first flapwise mode, which was $9 \%$ lower than the experimental result. However, the addition of the shaker and bungee cords to the NASTRAN model raised the first frequency to within $3 \%$ of the test average. The further addition of gravity brought the first flapwise frequency to within $1.3 \%$ of the test result. These modifications to the model had a minimal effect ( $<0.1 \%$ ) on the higher frequencies. The NASTRAN results in figure 12 and in table 2 include the boundary conditions and gravity.

## Instrumented Blades-Analysis and Experiment

The NASTRAN model of the pressure-instrumented blade also predicted the shake test results. Analysis correctly determined that the pressure blade had $2.4 \%$ lower frequencies than production blades. Figure 13 compares the shake test results and the NASTRAN predictions for both the production and pressure-instrumented blades. The first two columns show the frequency differences between the production and pressure blades measured in the shake test. The last two columns show the NASTRAN calculations for those blades. These results include the boundary conditions and gravity.

The strain-gage blade test results also correlated well with the analytical model. As mentioned in the NASTRAN Finite Element Models section, the NASTRAN production-blade model was used to predict the strain-gage blade frequencies. The 10 frequencies were from 0-2.3\% different from the NASTRAN frequencies.

## CONCLUSIONS

The shake test was effective in determining the dynamic similarity of the UH-60A blades flight tested in the Airloads Flight Research Phase of the Modern Technology Rotor Program. Production, strain-gage, and pressure-instrumented blades were tested to measure their frequencies and mode shapes, and analytic models were verified with the results.

## Shake Test

From the shake test of the instrumented blades, it was determined that their frequencies were slightly lower than those of production blades. The strain gage blade was less than $1 \%$ lower. The pressure blade had lower frequencies than any of the other five blades with a typical variation of $2-4 \%$ from the production blades. A comparison of the mode shapes revealed that the instrumented and production blades had nearly identical dynamic properties.

The analyses of the four standard blades established the production tolerance of modal frequencies. There was a measurable variability between the frequencies of production blades up to $2.4 \%$. It was demonstrated that this was not a result of variance in testing procedure by remeasuring one of the blades. Given the large variance among production blade frequencies, the $2-4 \%$ lower pressure blade frequencies were considered as only slightly lower. The strain-gage blade deviated less from the production average than did some of the standard blades.

A benefit from the shake test was the ability to choose the blades which best matched one another to conduct the airloads experiment. As a result, the two production blades with lower natural frequencies (blades 2 and 3 in fig. 10) were chosen for flight testing with the straingage and pressure-instrumented blades.

## Analysis

The finite element models were validated with the experimental results. The NASTRAN model of the production blade had excellent correlation with experimental data with a typical error from 0.1 to $2.4 \%$. The models of the instrumented blades had good correlation with the experiment. The pressure-instrumented-blade model was less than $3.2 \%$ different from the test with the exception of the fourth flapwise mode. The NASTRAN model of the production blade was sufficient for correlation with the strain-gage blade. The calculated frequencies correlated with less than $2.3 \%$ deviation from the strain-gage-blade test data.

The NASTRAN model of the pressure-instrumented blade predicted the systematically lower natural frequencies that were measured in the shake test. Therefore, the structural differences of the blade provided by Sikorsky were accurate.

It was important to model the boundary conditions of the shake test in the NASTRAN code. By adding the effects of the shaker, bungee cords, and gravity, the correlation of the first flapwise mode was significantly improved. This analysis illustrated the large effect the test conditions had on the lowest natural frequency.

The dynamic analysis was especially important because it verified that the structural information used to create the NASTRAN model was a correct blade representation. The same structural properties are used as an input to other rotorcraft analysis codes. Therefore, validating the structural information gives a known starting point to do other analyses. Since most codes assume equal blades, the degree of dynamic similarity among blades has been defined.

Ames Research Center<br>National Aeronautics and Space Administration Moffett Field, CA 94035, August 28, 1990

## APPENDIX A: NASTRAN CODE FOR UH-60A PRODUCTION BLADE




```
=, *1, =, .26,*.268333, ==
=,*1'=, '34,*.268333, =*
=,*1, =, .38,*.268333, == 
=49
=, *1, =, .44, *.268333 ==
=, *1, =, .448,*.268333, ==
=, *1, =, .455, *.268333, ==
=,*1,=,.465,*.268333,==
=, *1, =, .4833,*.268333, ==
=9
=, *1, =, .47, *.268333, =
=, *1, =, 46, *.268333, =
#, *1, =, .45, *.268333, ==
=, *1, =, .44,*.268333, == 
=5
GRID, *1, , *.097665, *.268333, ==
$6
CBEAM's show the 2 connected grids and the reference pt. location
which indicates blade twist by changing the direction of the principle
$ plane of bending
CBEAM, 9, 9, 9, 10, .3399, 0., 1.
= =
CBEAM, 14, 14, 14, 15,.381, 0., 1.
CBEAM, 15, 15, 15, 16, .424, 0., 1.
CBEAM, 16, 16, 16, 17, .462, 0., 1.
CBEAM, 17, 17, 17, 18, .498, 0., 1.
CBEAM, 18, 18, 18, 19, .537, 0., 1.
CBEAM, 19, 19, 19, 20, .5875, 0.., 1.
CBEAM, 20, 20, 20, 21, .5835, 0., 1.
CBEAM, 21, 21, 21, 22, .5815, 0., 1.
CBEAM, 23, 23, 23, 24, .5775, 0., 1.
CBEAM, 24,, 24, 24, 25, .5735,, 0., 1.
CBEAM, 25, 25, 25, 26, .5715, 0., 1
CBEAM, 26, 26, 26, 27, .5675, 0., 1.
CBEAM, 27, 27, 27, 28, .5645, 0., 1
CBEAM, 29, 29, 29, 30, .5585,, 0., 1.
CBEAM, 30, 30, 30, 31, .5565,'0.,, 1.
CBEAM, 31, 31, 31, 32, .5535, 0., 1.
CBEAM, 32, 32, 32, 33, .5515, 0., 1.
CBEAM, 33, 33, 33, 34, .5485, 0., 1.
CBEAM, 34, 34, 34, 35,.5445, 0., 1.
CBEAM, 36, 36, 36, 37, .5395, 0., 1.
CBEAM, 37, 37, 37, 38, .5355, 0., 1.
CBEAM, 38, 38, 38, 39, .5340, 0., 1.
CBEAM, 39, 39, 39, 40, .5305, 0., 1.
CBEAM, 40, 40, 40, 41, .5285, 0., 1.
```



```
=,*1,*1,*1,*1, .4835,==
$'PBEAM FORMAT
$PBEAM, P1O,MID,A,I1, I2, I12, J, NSM, +P2 (NOT (FOR GRID A)
$+P3, SO, X/XB, A, I1, 12, II2, J,NSM, +P4 (FOR GRID B=X/XB)
$+PQ', C1, C2, D1, D2,' E1, E2, F1, F2, +P5 CN (NOT USED)
$+P5, K1, K2, S1, S2,NSI(A),NSI(B), CW, CH, +P6 (NSI=MASS INERTIA ABOUT CG)
$
PBEAM, 9, 1,.00953, 2.66E-4, 2.69E-2, , .547E-3, 0.847, +92
+92, ('SA, +93
+93, YESA, 1.0,0,00619, , , , , , +95
+95, , , , , 0.0184
PBEAM, 10, 1,.0163, 2.66E-4,.114E-2, , .547E-3, 0.625, +103
+103, YESA, 1.0,.0148, , , , , +105
+105, , , , , 0.0184
$
PBEAM, 11, 1, .0150, 2.83E-4, .114E-2, , .388E-3, 0.625, +113
+113, YESA, 1.0, .0157, , , , , +115
+115, , , , 0.0184
$
PBEAM, 12, 1, .0157, 2.83E-4, .114E-2, . 388E-3, 0.625, +123
+125, , , , , 0.0184
$
PBEAM, 13, 1, .0165, 2.83E-4, .114E-2, , .388E-3, 0.625, +133
+133, YESA, 1.0,.0178, , , ', ', +135
+135,', , , 0.0184, , , , +136
+136, , , , .00403
```

```
\(\$\)
PBEAM, \(14,1, .0178,2.83 E-4, .114 \mathrm{E}-2, \quad .388 \mathrm{E}-3,0.625,+143\)
+143, YESA, \(1.0, .0170,, 1+146,+145\)
\(\begin{array}{ll}+145, \\ +146, & , .001403,\end{array}, .0064 \mathrm{I}\)
PBEAM, 15, 1, .0181, .833E-4, .141E-2, , 388E-3, 0.21492, +153
+153,' YESA, 1.0, .0173, , , \(156,+155\)
```



```
\$ PBEAM, 16, 1, . \(0173, .833 \mathrm{E}-4, .141 \mathrm{E}-2, \quad .388 \mathrm{E}-3,0.21492,+163\)
+163, YESÁ, 1.0, .0165, , , +16'́, +165
+165, , , \(0.024,, ~, ~ '+166\)
+166, , , .0 10107, , . 0134
PBEAM, \(17,1, .0165, .833 E-4, .141 \mathrm{E}-2, .388 \mathrm{E}-3,0.21492,+173\)
PBEAM, YESA, 1. \(0, .0158,173\), , \(, 1,175\)
+175,
+176,, , \(.0134,0,0.0174\)
FBEAM, \(18,1, .0158, .833 \mathrm{E}-4, .141 \mathrm{E}-2\), , 202E-3, \(0.21492,+183\)
+183 , YESA, 1.0, .0148, , , \(186^{\prime},+185\)
\(+185, \quad, \quad 0,024, \prime, 186\)
+186', ' . .01́7', , . \(020{ }^{\prime}\)
PBEAM, \(19,1, .0148, .833 E-4, .141 \mathrm{E}-2, .202 \mathrm{E}-3,0.21492,+193\)
PBEAM, \(19,{ }^{1} .0148,833 E-4, .141 E-2\),
+193, YESA, \(1.0,{ }_{2} .0138, \quad, \quad, 195\)
```



```
PBEAM, 20, 1, .0138, .833E-4, .141E-2, , .202E-3, 0.21492, +203
```




```
\(\$\)
PBEAM, 21, \(1, .0128, .833 \mathrm{E}-4, .141 \mathrm{E}-2, .202 \mathrm{E}-3,0.21492,+213\)
+213,' YESÁ, 1.0, .0120, , ' +215 , +215
+215, , , . 024, , +216
+216, , '. \(03355^{.024}, .0403\)
PBEAM, 22, \(1 ; .0120, .833 \mathrm{E}-4, .141 \mathrm{E}-2\), , \(202 \mathrm{E}-3,0.21492,+223\)
+223, YESA, \(1.0, .0112\), ,,+225
```



```
+226, , . 0403 , , .0470'
\(\$\)
PBEAM, 23, \(1, .0193, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), , 202E-3, \(0.21292,+233\)
+233, YESA, \(1.0, .0194\), ,,+235
+235, , ', '. 0318 ,, , ' +236
+236', , '.0470', , . \(0533^{\prime}\)
\$
PBEAM, 24, 1, . \(0194, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), , \(202 \mathrm{E}-3,0.21292,+243\)
+243, YESA, 1.0, 01945, , , , +245'
\(+245,1,1, .0318,1,1+246\)
+246, , . . \(0537^{\prime}, ~, .0604\)
\(\$\)
PBEAM, \(25,1, .01945, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), . \(189 \mathrm{E}-3,0.21292,+253\)
+253, YESA, \(1.0, .0195,1,156,+255\)
+255, , , . \(0318,1,1+256\)
+256, , . 0604 , , . 0685
\$
PBEAM, 26, 1, .0195, .668E-4, .252E-2, , . 189E-3, 0.21292, +263
+263, YESA, 1.0, , , , +265
+265, , , . 0318, , ', ' ' +266
+266,' , '. 068 '
\$
PBEAM, 27, 1, .0195, .668E-4, .252E-2, , .189E-3, 0.21292, +273
+273, YESA, 1.0, , , , +275
```



```
+276, , . 0685
PBEAM, 28, 1, .0195, .668E-4, .252E-2, , .189E-3, 0.21192, +283
```

```
+283, YESA, l.0, 0196, , ' +28', ' +285
+285,, , , , .0318, , , , '286
+286,', .0685
$
PBEAM, 29, 1, .0196,668E-4, .252E-2 , .189E-3, 0.21192, +293
+293, YESA, 1.0, .0197, , , , , +295
+295, , . .' %',
PBEAM, 30, 1, .0197, .668E-4, .252E-2, , .189E-3, 0.21192, +303
+303,' YESA, 1.0, , , , , +305
+305', , ', .0318', ', ', ' +306
+306', '. .0685
PBEAM, 31, I, .0197, .668E-4, .252E-2, .189E-3, 0.21192, +313
+313, YESA, 1.0, 0.0198, , '+'31''
+316', '.0685
$
PBEAM, 32, 1, .0198, . 668E-4, .252E-2, , . 189E-3, 0.21192, +323
+323, YESA, 1.0, 0199, , , ', ', +325
+325', , ,, .03i8, , , , '+326
+326, ,. .0685
$
PBEAM, 33, 1, .0199, .668E-4, .252E-2, , .189E-3, 0.21192, +333
+333, YESÁ, 1.0, , , , , +335
+335,', '0685.0318, , , +336
+336, ,.0685
$
PBEAM, 34, 1, .0199, .668E-4, .252E-2, .189E-3, 0.21292, +343
+343, YESA, 1.0, 0200, , , , , +345
+345, , ,
+346, , .0685
PBEAM, 35, 1, .0200,.668E-4, .252E-2, , .189E-3, 0.21292, +353
+353, YESA, 1.0, , , , , +355
+355, , , '055.0318, , , +356
+356', , '.0685
$
PBEAM, 36, 1,.0200,.668E-4, .252E-2, , .189E-3, 0.21292, +363
+363, YESA, 1.0, 0201, , '+366, +365
+365, , , , .03i8, , , , '+366
+366,' ', .0685
$
PBEAM, 37, 1, .0201, ,668E-4, .252E-2, . 189E-3, 0.21292, +373
+375, , .068'5
$
PBEAM, 38, 1, .0202, .668E-4, .252E-2, , .189E-3, 0.21292, +383
+383, YESA, i.0, , , , + +385
+385', , '0' '.0318', ', ', ' +386
+386, , . 0685
$
PBEAM, 39, 1, .0202,.668E-4, .252E-2, , .189E-3, 0.21292, +393
+393, YESA, 1.0, .0203, , , , , +395
+395, , '0685.0318, , , , '+396
+396, . .0685
$
PBEAM, 40, 1,.0203, .668E-4, .252E-2, , .189E-3, 0.21292, +403
+403,' YESA, '1.0, , , , , +405
+405, , ', .0318', ', ' ' '+406
+406',' '.0685
$
PBEAM, 41, 1,.0203, .668E-4, .252E-2, , .189E-3, 0.21292, +413
PBEAM, YESA, 1.0,0, 1.03, .668E-4, +415
```



```
+416', ', .0685
$
PBEAM, 42, 1, .0203,.668E-4,.252E-2, , .189E-3, 0.21292, +423
+423, YESA, 1.0, , , , +425
+425, , , , .0318', , , ' '+426
```

```
+426, , . 0685
$
PBEAM, 43, 1,.0203, .668E-4,.252E-2, , .189E-3, 0.21292, +433
+433, YESA, 1.0, , , , +435
+435, , ,0'85 .0318, , , , +436
+436', '..0685
PBEAM, 44, 1,.0203,.668E-4,.252E-2, ,.189E-3, 0.21292, +443
PBEAM, 44, 1,.0203, .668E-4, . 255
+445,' , ', .0318', ', ', ' '+446
+446', ', .0685
PBEAM, 45, 1, .0203,.668E-4, .252E-2, , .192E-3, 0.21292, +453
+453, YESA, i.0, , , , , +455
+455, , ,', .03i8', ' ', ' '456
+456', '. . }068
$
PBEAM, 46, 1,.0203,.668E-4, .252E-2, , .192E-3, 0.21292, +463
+463, YESA, I.0, .018, , , , , +466
+466', '. .06'85
$
PBEAM, 47, 1, .0203,.668E-4, .252E-2, , .192E-3, 0.21292, +473
+473,'YESA, 1.0,', , , +475
+475, , , ', .03i8',' ', ', '476
+476', ,.0685, , .067i
PBEAM, 48, 1,.0203,.668E-4,.252E-2, ,.192E-3, 0.21292, +483
+483, YESA, 1.0, , , , +485
+485', , ', .0318',' ', ', '486
+486', , .0677i', , .0660
PBEAM, 49, 1,.0203,.694E-4,.252E-2, , .192E-3, 0.22092, +493
+493, YESA, i.0., ', ', , , +495
+495, , , , .03474,', , , +496
+496', ' .0660', , .06'44
$
PBEAM, 50, 1, .0203,.694E-4, .253E-2, ,.192E-3, 0.22092, +503
+503,'YESA, 1.0.0,
+506,', ..0644', , .0633
$
PBEAM, 51, 1, .0203, .694E-4, .253E-2, , .192E-3, 0.22092, +513
+513,' YESA, 1.0,
+516', ' .0633, ,.0617'
PBEAM, 52, 1,.0203,.694E-4, .253E-2, , .193E-3, 0.22092, +523
PEEAM, 52, 1,.0203, .694E-4, . 253
+525', ,', ,.03474',',',', +526
+525', '.0617, , .0609'
$BEAM, 53, 1,.0203, .694E-4, .253E-2, , .193E-3, 0.22092, +533
+533,' YESA, 1.0, , ', ', ', ', ,'5354
+535,', '.0609, , .03474'050'
$PBEAM, 54, 1,.0203, .694E-4, .253E-2, , .193E-3, 0.21592, +543
+543, YESA, 1.0,0'Á'A, , , , +545
+545, , ,',03474,', , +546
+546', ', '0590', , .0585'
$
PBEAM, 55, 1,.0203,.694E-4,.253E-2, , .193E-3, 0.21592, +553
```



```
+556', , .0585, , .0569
$
PBEAM, 56, 1, .0203,.694E-4, .253E-2, , .193E-3, 0.21592, +563
+563, YESA, 1.0 , %'Á'A, , , +565
+563, YESA, 1.0.03'474,',', ',+566
+566,', .0569', , .0563'
$
```

PBEAM, $57,{ }^{1} .0203, .694 E-4, .253 E-2, .193 E-3,0.21592,+573$

+576', ' . 0563 3, , . $0547^{\prime}$
\$
PBEAM, 58, $1, .0203, .694 E-4, .253 E-2, ~ .193 E-3,0.21592,+583$
+583, YESA, 1.0 ,,+585
+585', , , ' . $03474 \mathbf{4}^{\prime}, ', ',+586$
+586', ' '. 0547, , . 0510
PBEAM, 59, $1, .0203, .694 E-4, .253 E-2, ~ .193 E-3,0.21592,+593$
PBEAM, $59,1.0203, .694 E-4,+25$
+593, YESA, $1.0,1,1,155$
$+595,, .0510^{, 03474,}, \quad .0478^{\prime},+596$
\$
PBEAM, 60, $1, .0203, .699 E-4, .243 E-2,, .193 E-3,0.21892,+603$

+605, , , , . $03474,1,1+606$
+606, , . 0478 , , . $0451^{\prime}$
\$
PBEAM, 61, $1, .0203, .699 E-4, .243 E-2, ~ .193 E-3,0.21892,+613$
$\begin{aligned} & \text { +613, YESA, } 1.0 \\ & +615,\end{aligned}, 1+615$
+616', ' '.0451, , . 0416
\$
PBEAM, 62, $1, .0203, .699 E-4, .243 \mathrm{E}-2, ~ .193 \mathrm{E}-3,0.21892$, +623
+623. YESA, 1.0, , , , +625
+625', , , . 03474 A $^{\prime}, ', '+626$
+626', ,. .0416, , .0390'
$\$$
PBEAM, 63, $1, .0203, .699 E-4, .243 \mathrm{E}-2$, , $193 \mathrm{E}-3,0.21892,+633$
+633, YESÁ, i.0, , , , +635
+635, , , 03474, , , +636
+636', , . 0390 , , . 0419
\$
PBEAM, 64, $1, .0203, .699 \mathrm{E}-4, .243 \mathrm{E}-2$, , $193 \mathrm{E}-3,0.21892,+643$
+643, YESÁ, 1.0, , , +645
+645, , , ' .03474 ' $^{\prime}, ~, ~ ', ~ '+646 ~$
+646', ' . . 04119 , , . 0456
PBEAM, $65,1, .0203, .707 E-4, .210 E-2, \quad .193 E-3,0.21892,+653$
+653, YESA, 1.0 , , , , +655
+655 , , '045 . 03474, , , +656
+656', ' . '0456', , .0499'
\$
PBEAM, 66, $1, .0203, .707 E-4, .210 E-2$, , 193E-3, $0.24192,+663$


$\$$
PBEAM, $67,1, .0203, .707 E-4, .210 \mathrm{E}-2$, , $193 \mathrm{E}-3,0.24192,+673$
+673, YESA, 1.0, $, ~, ~,+675$
+675', , , .034́74, ' ' , ' + +676
+676, , . 0537, , 0582
\$
PBEAM, 68, 1, .0203, .707E-4, .210E-2, , .193E-3, 0.24192, +683
+683, YESA, i.0, , , , +685
+685, , , , . $03971,,,+686$
+686, , . 0582 , , . 0456
\$
PBEAM, 69, $1, .0203, .707 E-4, .210 \mathrm{E}-2,, .193 \mathrm{E}-3,0.24192,+693$
+693, YESA, 1.0 . 0 , +695
+695', , , . 03971 ', ' , ' '+696
+696', ' . 04 ' $^{\prime} 6$, , . 03005
+
PBEAM, $70,1, .0203, .707 E-4, .210 E-2, ~ .193 E-3,0.24192,+703$ +703, YESA, $1.0,1,1,+705$
+705, , ' $, 03971, ', ',+706$
+706', , . 030005, , . 0161
\$
PBEAM, $71,1, .0203, .707 \mathrm{E}-4, .215 \mathrm{E}-2,1.193 \mathrm{E}-3,0.27392,+713$ +713, YESA, 1.0, , , , , +715

```
+715, , , 04650 , , +716
+716, ,.0161, , . 00268
PBEAM, 72, 1, .0203, .707E-4, .215E-2, , .193E-3, 0.27392, +723
+723, YESA, 1.0, , , , +725
+725, , , \(04650,{ }^{\prime},+726\)
+726', ' . .002 68 , , -.010'
\$
PBEAM, 73, 1,.0203, .707E-4, .215E-2, , .193E-3, 0.27392, +733
+733, YESA, 1.0,
+735, , , \(.04650, \prime,+736\)
+736', ', -.0́10̀7, , -.02'45
\$
PBEAM, 74, 1, .0203, .707E-4, .215E-2, , .193E-3, 0.27392, +743
+743, YESA, 1.0 , 1 , , , +745
+745 ', , ' \(.04650,{ }^{\prime}, ', '+746\)
+746, , -.0245, , -. 0247
\$
PBEAM, \(75,1, .0203, .707 E-4, .215 E-2, ~ .193 E-3,0.27392,+753\)
+753, YESÁ, 1.0,, , +755
+755, , \(.04354, ~,+756\)
+756', ', -.02477, , -.0249'
\(\$\)
PBEAM, \(76,1, .0203, .707 \mathrm{E}-4, .210 \mathrm{E}-2, ~ .193 \mathrm{E}-3,0.27292,+763\)
+763, YESA, \(1.0,1,+765\)
+765, , , . \(04354, \prime, 1+766\)
+766', ' '-.0́2'49, , -.0251'
\$
PBEAM, 77, \(1, .0203, .707 E-4, .210 \mathrm{E}-2, \quad .193 \mathrm{E}-3,0.27292,+773\)
+773, YESA, \(1.0 .0431^{\prime}, ~, ~, ~+775\)
+775, , , . \(04354,{ }^{\prime},+776\)
+776, ', -. \(0251^{\circ}, 1,-.0252^{\prime}\)
\$
PBEAM, 78, 1, .0203, .707E-4, .210E-2, , .193E-3, 0.27292, +783
+783. YESA, 1.0, \(1 .,+785\)
+785', , , \(.04354, ', ', '+786\)
+786', ', -.0́252, , -. \(0253^{\prime}\)
\$
PBEAM, 79, \(1, .0203, .707 \mathrm{E}-4, .210 \mathrm{E}-2, ~ .192 \mathrm{E}-3,0.27292,+793\)
+793, YESA, 1.0, \(, 1,+795\)
```



```
+796, ', '. \(0255^{\circ}\), , -.0254
\$
PBEAM, \(80,1, .0203, .707 \mathrm{E}-4, .210 \mathrm{E}-2\), , \(192 \mathrm{E}-3,0.27292,+803\)
+803, YESÁ, \(1.0,1,+805\)
+805, , , \(04354,1,1+806\)
+806,' , '..0254, , -.0255'
\$
PBEAM, 81, \(1, .0203, .686 \mathrm{E}-4, .198 \mathrm{E}-2, \quad .192 \mathrm{E}-3,0.27292,+813\)
+813 , YESA, \(1.0,1,1,+815\)
+815', , , ' \(.04354^{\prime}, ~ ' ~ ', ~ '+816 ~\)
+816, ', -. \(02555^{\prime}, ~, ~-.02565\)
\(\$\)
PBEAM, 82, 1, .0203, .686E-4, .198E-2, , .192E-3, 0.27092, +823
+823, YESA, 1.0
+825', , ' \(, .04354^{\prime}, ', ',+826\)
+826', ', '. \(025655^{\prime}, ~-.0258\)
\$
PBEAM, 83, 1, .0203, .686E-4, .198E-2, , .192E-3, 0.27092, +833
+833, YESA, 1.0, , , , +835
+835', , , ' 0435 ' \(^{\prime}\) ' ' ' , ' +836
+836', ' '-.0. \(25^{\prime} 8^{\circ}\), , -. \(0268^{\prime}\)
\(\$\)
PBEAM, \(84,1, .0203, .686 \mathrm{E}-4, .198 \mathrm{E}-2,, .192 \mathrm{E}-3,0.27092,+843\)
+843, YESA, 1.0, \(1.1+845\)
+845, , , \(.04354,{ }^{\prime},{ }^{\prime}, '+846\)
+846,' , '-.0268, , -.02 \({ }^{\prime} 3{ }^{\prime}\)
\$
PBEAM, 85, 1, .0203, .686E-4, .198E-2, , .192E-3, 0.27092, +853
+853, YESÁ, 1.0, \(1 ., 1+855\)
```



```
+856, , '. '. 0273 ,,\(-.0279^{\prime}\)
```

```
$
PBEAM, 86, 1,.0203, .673E-4,. 156E-2, , .189E-3, 0.33892, +863
+863,' YESA, 1.0, +, , , , , %65
+865, , , '.03321, ', +866
+866,', -.0279, , -.0287'
PBEAM, 87, 1,.0203,.673E-4,.156E-2, ,.189E-3, 0.33892, +873
+873, YESA, 1.0,
+875, , '.0287, , -.03321,',
$
PBEAM, 88, 1,.0203,.673E-4,.156E-2, , .189E-3, 0.33892, +883
+883, YESA, 1.0,
+885, , , -.0307, , -.0671
$PEAM, 89, 1,.0203,.673E-4,.156E-2, , .189E-3, 0.33892, +893
```



```
+895, 1 '.0671, . , -. }10
$BEAM, 90, 1, .0203,.599E-4,.141E-2, ,.189E-3, 0.37092, +903
+903, YESA, 1.0, % , , , +005
+905, , , '05861,', , +906
+906, ,' -.105, , -. 1369
FBEAM, 91, 1,.0203,.599E-4,.141E-2, ,.189E-3, 0.37092, +913
+913, YESA, 1.0, 隹, , , , +915
+915, , , .05861, , +916
+916', ',.1369, , -.1749'
PBEAM, 92, 1,.0203, .599E-4, .141E-2, ,.189E-3, 0.37092, +923
+923,' YESÁ, 1.0 , %'&61, , , , +925
+925,', ,',.05861' ',', ' '+926
+926, ', '.1749, , -.1342'
$BEAM, 93, 1,.0203, .599E-4, .141E-2, , .189E-3, 0.37092, +933
+933, YESA, 1.0,05i23', , , +935
+935', , , '.05723', ' ', ' , '936
+936, ,':.1342, , -.0805'
$
PBEAM, 94, 1,.0203,.447E-4,.117E-2, ,.189E-3, 0.34892, +943
+943, YESA, 1.0, %123, , , ' +945
+946, , -.0805, , -.0322
$
PBEAM, 95, 1,.0203,.447E-4,.117E-2, , .189E-3, 0.34892, +953
+953, YESA, 1.0, , , , , +955
+955', , , , .05723,', , , +956
+955, , '.0322, , .0832
$
PBEAM, 96, 1,.0203, .447E-4,.117E-2, , .189E-3, 0.26092, +963
+963, YESA, i.0, 缺, , , +965
+965', , , , .03805,',',' +966
+966, , '.0832, , . }185
$
PBEAM, 97, 1,.0203,.269E-4,.118E-2, ,.189E-3, 0.26092, +973
+973, YESÁ, 1.0., , , , +975
+975', , ', .03805,' ' ', '+976
+976', ', '.1852,', . 2651
$
PBEAM, 98, 1,.0203,.269E-4,.118E-2, ,.189E-3,0.26092, +983
PBEAM, 98, 1,.0203, .269E-4, . . 11 
+983,' VESA, 1.0.03'0',', , ' +9885
+985, , '. '651, . . . 4773
$
PBEAM, 99, 1,.0203,.221E-4,.134E-2, ,.189E-3, 0.08292, +993
+993, YESA, 1.0, O, , 2LIE-4, +995
+995, , , ', .03805,',',','+996
+996,', ..4773', , .4773'
$
ENDDATA
```


# APPENDIX B: NASTRAN CODE FOR UH-60A PRESSURE-INSTRUMENTED BLADE 



```
THE PRESSURE INSTRUMENTED BLADE MODEL WAS CREATED BY MODIFYING
THE PROOUCTION BLADE MODEL. THE CHANGES WERE MADE BASED ON INFORMATION
PROVIDED BY SIKORSKY AIRCRAFT. THE ONLY CHANGES WERE MADE TO THE
\$ PBEAM CARDS WHICH ARE SHOWN BELOW.
\$ THE NASTRAN PROGRAM FOR THE PRESSURE INSTRUMENTED BLADE DIFFERS FROM
\$ THE PRODUCTION BLADE IN TERMS OF 4 STRUCTURAL PARAMETERS:
    1. MASS/UNIT LENGTH, (NSM)
    2. MASS C.G. (M1, M2)
    3. FLAPWISE STIFFNESS (11)
    4. CHORDWISE STIFFNESS, (12)
*******************
PBEAM \(9,1, .00953,2.66 \mathrm{E}-4,2.69 \mathrm{E}-2,1.547 \mathrm{E}-3,0.559,+93\)
+93, YESA, 1.0, . \(00619,1,1,1+95\)
+95, , , 0.0184
\(\$\)
PBEAM, \(10,1, .0163,2.66 E-4, .114 \mathrm{E}-2, \quad .547 \mathrm{E}-3,0.637,+103\)
+103, YESA, \(1.0, .0148, \ldots,,+105\)
+105, , , , 0.0184
\$
PBEAM, 11, 1, . \(0150,2.83 E-4, .114 E-2, \quad .388 E-3,0.637,+113\)
+113, YESA, \(1.0, .0157,1,1,+115\)
+115, , , , 0.0184
\$
PBEAM, 12, \(1, .0157,2.83 E-4, .114 \mathrm{E}-2, \quad .388 \mathrm{E}-3,0.637,+123\)
+123 , YESA, 1.0, .0165, , , , +125
\(+125, ~, ~, ~ 0.0184,,,+126\)
+126, , , ', '. 0161
\(\$\)
PBEAM, 13, 1, . \(0165,2.83 \mathrm{E}-4, .114 \mathrm{E}-2, \quad .388 \mathrm{E}-3,0.637,+133\)
+133 , YESA, 1.0, .0178, , , , +135
+135, YESA, \(1.01 .0178, \quad, \quad\) ' \(+136^{\prime}\)
+136', , -. 0 '16́ \(, ~,-.0134\)
\(\$\)
PBEAM, 14, \(1, .0178,2.83 E-4, .114 E-2, \quad .388 \mathrm{E}-3,0.637,+143\)
+143 , YESA, 1.0, .0170, , , +145
+145, , , 0.024, , ' +146
```



```
\(\$\)
PBEAM, 15, 1, .0181, .833E-4, .141E-2, , .388E-3, 0.217, +153
+153 , YESA, 1.0,.0173, , , +155
+155, , \(0.024,1,{ }^{\prime},+156\)
+156, , '. '010́7, , -0.006'0
\(\$\)
PBEAM, \(16,1_{1} .0173, .833 E-4, .141 E-2, .388 E-3,0.217,+163\)
+163, YESA, 1.0, .0165, , , , +165
\(+165, \quad\), \(0.024, \quad+166\)
```



```
\$
PBEAM, 17, 1, .0165, .833E-4, .141E-2, \(.388 \mathrm{E}-3,0.217,+173\)
+173, YESÁ, 1.0, .0158, , , , +175
\(+175,1,1,0.024,1,1+176\)
+176', ' - 0.00027 , , \(0.0^{\prime}\)
PBEAM, \(18,1, .0158, .833 E-4, .141 \mathrm{E}-2,, .202 \mathrm{E}-3,0.217,+183\)
+183, YESA, \(1.0, .0148,1,1,+185\)
+185, , , , \(0.024,1, ~, ~+186\)
+186', ', \(0 . \dot{0}^{\prime}, \quad, 0.0027^{\prime}\)
\$
PBEAM, 19, 1, .0148, .833E-4, .141E-2, , .202E-3, 0.217, +193
+193 , YESA, \(1.0, .0138,1,1,+195\)
```



```
+196, , ' \(0.00054, \quad 0.0027\)
\(\$\)
PBEAM, 20 1, . \(0138, .833 E-4, .141 \mathrm{E}-2\), , 202E-3, \(0.217,+203\)
+203, YESA, \(1.0, .0128\), ,,+205
+205', ,, .02 A \(^{\prime}, \quad\) ' \(+206^{\prime}\)
\(\begin{aligned} & +205, \\ & +206,\end{aligned}, \dot{0} .0027^{.024}, 0.0080^{\prime}\)
```

```
PBEAM, 21, \(1, .0128, .833 \mathrm{E}-4, .141 \mathrm{E}-2, .202 \mathrm{E}-3,0.217,+213\)
+213, YESA, \(1.0, .0120,,+21 \sigma^{\prime},+215\)
```



```
\$
PBEAM, 22, \(1, .0120, .833 \mathrm{E}-4, .141 \mathrm{E}-2, .202 \mathrm{E}-3,0.215,+223\)
+223 , YESA, \(1.0, .0112,\), ,,+225
+225, , '.0147, . \(024, .0215^{\prime}\)
\(\$\)
PBEAM, 23, \(1, .0193, .668 \mathrm{E}-4, .252 \mathrm{E}-2, .202 \mathrm{E}-3,0.215,+233\)
+233 , YESA, \(1.0,0194\), , , +235
+235, , , , .0318, , , +236
+236, , '. 02 15, , . 0268
\(\$\)
PBEAM, 24, \(1, .0194, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), , 202E-3, \(0.215,+243\)
+243, YESA, \(1.0, .01945, \quad, \quad, 1,+245^{\prime}\)
+245, , , . 0318 , , +246
+246', ', .02́68, , . \(033{ }^{\prime}\)
\$
PBEAM, 25, 1, .01945, . 668E-4, .252E-2, , .189E-3, 0.215, +253
+253, YESA, \(1.0, .0195, \quad, \quad+256^{\prime},+255\)
+255, , '0 \(0318,{ }^{\prime}, \quad+256\)
+256, ', . \(0335^{\prime}\), , . 0402
\(\$\)
PBEAM, 26, \(1, .0195, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), , \(189 \mathrm{E}-3,0.215,+263\)
+263 , YESA, 1.0, \(, 1,+265\)
+265, , , '. 0318 ' ' ', ' ' +266
+266', ', . 0402 , , . 0408
\(\$\)
PBEAM, 27, \(1, .0195, .668 \mathrm{E}-4, .252 \mathrm{E}-2,, .189 \mathrm{E}-3,0.215,+273\)
+273, YESA, \(1.0,10^{\prime}, 1,+275\)
+275, , , , . \(0318^{\prime}, 1,1,1+276\)
+276 , , . 0408
\$
PBEAM, 28, 1, .0195,.668E-4, .252E-2, , .189E-3, 0.215, +283
+283 , YESA, \(1.0, .0196\), , , +285
+285, , , .0318, , , , +286
+286', ', .0408
\$PEAM, 29, \(1, .0196, .668 \mathrm{E}-4, .252 \mathrm{E}-2,1.189 \mathrm{E}-3,0.2204,+293\)
+293 , YESA, 1.0, 0197, , , , , +295
+295, , , . \(0318, ~, ~, ~+296\)
+296', '. 0408
PBEAM, \(30,1, .0197, .668 \mathrm{E}-4, .252 \mathrm{E}-2, \quad, .189 \mathrm{E}-3,0.2204,+303\)
+303, YESA, \(1.0,1,+305\)
+305, , , ó .0318, ' ' ' ' ' +306
+306', : . 0408
\(\$\)
PBEAM, 31, \(1, .0197, .668 \mathrm{E}-4, .252 \mathrm{E}-2, . .189 \mathrm{E}-3,0.2204,+313\)
+313 , YESA, \(1.0,0198\), , ,,+315
+315, , , \(.0318,,,,+316\)
+316, , . 0408
\$
PBEAM, 32, \(1, .0198, .668 \mathrm{E}-4, .252 \mathrm{E}-2\), . 189E-3, \(0.2204,+323\)
+323, YESA, 1.0, 0199 , , , ' 226 , +325
\(+325^{\prime}, ~, ~, ~ .0318, ~, ~, ~+326\)
+326, , . . 04088
\(\$\)
PBEAM, 33, 1, .0199, .668E-4, .252E-2, , .189E-3, 0.2204, +333
+333, YESA, 1.0, , , +335
+335, , , , .0318', ', ' ' ' + 336
+336,', '. 0408
PBEAM, \(34,1, .0199,668 \mathrm{E}-4, .252 \mathrm{E}-2\), , \(189 \mathrm{E}-3,0.2204,+343\)
+343 , YESA, 1.0, .0200, , , +345
+345, , , .0318, , , +346
+346 , ' '. 0408
PBEAM, 35, 1,.0200, .668E-4, .252E-2, , .189E-3, \(0.2214,+353\)
```

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+353, YESA, 1.0, %', , , ' +35655
+355, , ', .0318', ', ', +356
+356, . . }040
PBEAM, 36, 1, .0200, .668E-4, .252E-2, . 189E-3, 0.2214, +363
+363, YESA, 1.0, .020i, , , , +365
+365, , , , .0318, , , , +366
+366', , .0408
$
PBEAM, 37, 1,.0201,.668E-4, .252E-2, .189E-3, 0.2214, +373
```



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+376', ',.0408
$
PBEAM, 38, 1,.0202,.668E-4,.252E-2, ,.189E-3, 0.2214, +383
+383,' YESA, 1.0, , , , , +385
+385, , , .0318, , , , +386
+386', ', .0408
$
PBEAM, 39, 1,.0202,.668E-4, .252E-2, . .189E-3, 0.2214, +393
+393, YESÁ, 1.0,.0203, , , , +395
+395, , , , .0318, , , +396
+396, . . }040
PBEAM, 40, 1,.0203,.668E-4, .252E-2, ,.189E-3, 0.2214, +403
+403, YESA, 1.0, , , , , +405
+405, , , .0318, , , , +406
+406', ', .0408
$PBEAM, 41, 1,.0203,.668E-4, .252E-2, ,.189E-3, 0.2214, +413
PBEAM, 41, 1, .0203, .668E-4, .252
+413, YESA, 1.0,0318', , , , +416
+416', '.0408
s
PBEAM, 42, 1,.0203,.668E-4,.252E-2, . .189E-3, 0.2214, +423
+423,'YESA, 1.0,
+425, , ,',
+426, , .0408
$
PBEAM, 43, 1,.0203,.668E-4, .252E-2, ,.189E-3, 0.2214, +433
+433, YESÁ, 1.0, , , , +435
+435, , , .0318', , , '+436
+436', ' .0408
$
PBEAM, 44, 1, .0203,.668E-4,.252E-2, ,.189E-3, 0.2214, +443
+443,' YESA, 1.0, , , , +445
+445, , , .0318', , , , +446
+446, , .0408
PBEAM, 45, 1,.0203,.668E-4, .252E-2, ,.192E-3, 0.2214, +453
PBEAM, 45, 1, .0203, .668E-4, .252
+453, YESA, 1.0. %'18, , , ', '+456
+456', , .0408
$
PBEAM, 46, 1,.0203,.668E-4, .252E-2, , .192E-3, 0.2214, +463
+463, YESÁ, 1.0, , , , , +465
+463, YESA, 1.0', '18,', ,', '+466
+466',',.0408
$
$BEAM, 47, 1,.0203,.668E-4, .252E-2, ,.192E-3, 0.2214, +473
+473, YESA, 1.0, , , , , +475
+475, YESA, 1.0.0318', ', ', ' '476
+476, , .0408, , . 0402
$
PBEAM, 48, 1,.0203,.668E-4,.252E-2, , .192E-3, 0.2214, +483
PBEAM, YESA, 1.0, .0, , 068E-4, +485
+485', , .03i8', ' ' '+486
+486', , '.0402, , . 0389
$
PBEAM, 49, 1, .0203,.694E-4, .252E-2, , .192E-3, 0.223, +493
+493, YESA, 1.0, , , , , +495
+495', , ,, ,.03474,',',',+496
```

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+496, , .0389, , . 0376
PBEAM, 50, 1,.0203,.694E-4,.253E-2, , .192E-3, 0.223, +503
+503, YESÁ, 1.0, , , , +505
+505, , , .03474',', ', ' +506
+506, ' .0483'3, , .0478
$
PBEAM, 51, 1,.0203,.694E-4, .253E-2, , .192E-3, 0.223, +513
+513, YESA, 1.0, % ' % , , , ' +515
+516, , .0478, , .0461
PBEAM, 52, 1, .0203,.694E-4, .253E-2, . 193E-3, 0.223, +523
PBEAM, 5ESA, 1.0,0, +523, YES, 隹, , , +525
+525, , ,',.03474',',',',+526
+526', , .0461, , .0448'
$BEAM, 53, 1,.0203, .694E-4, .253E-2, , .193E-3, 0.223, +533
FBEAM, 53, 1,.0203, .694E-4, .253
+535', , ,',03474',',',', '536
+536', ' . .0448, , .0435
$
PBEAM, 54, 1,.0203,.694E-4,.253E-2, , .193E-3, 0.223, +543
M, +543, YESA, 1.0,
```



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+546', ' ..0435,',.0424
PBEAM, 55, 1,.0203,.694E-4, .253E-2, , .193E-3, 0.218, +553
```



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+556, ',.0424, , .0408
$
PBEAM, 56, 1, .0203, .694E-4, .253E-2, , .193E-3, 0.218, +563
+563, YESA, 1.0, , , +565
+565', , ', .03474',',', +566
+566, ,.0408, , .0400
$
PBEAM, 57, 1, .0203, .694E-4, .253E-2, , .193E-3, 0.218, +573
+573, YESÁ, 1.0,03Á', ', ', '+575
+575,', '.0'0',.03474',',
$
PBEAM, 58, 1,.0203, .694E-4,.253E-2, , .193E-3, 0.218, +583
+583, YESA, 1.0,0, l, , +585
+585', , ',' .03474',',' ','+586
+586', ' ..0322', , .0295'
$
PBEAM, 59, 1, .0203, .694E-4, .253E-2, , .193E-3, 0.218, +593
+593,'YESA, 1.0,',',', ', +595
+596', ' .02955, , .0260'
$BEAM, 60, 1,.0203, .699E-4, .251E-2, , .193E-3, 0.221, +603
PBEAM, 60, 1, .0203,.699E-4, .25
+603, YESA, 1.0, ('1774,',', , +606
+606', , .0260,, , .0233
$
PBEAM, 61, 1,.0203, .699E-4,.251E-2, ,.193E-3, 0.221, +613
+613, YESA, 1.0,
+615, , , O3474, , , +616
+616, , .02333, , .0201
$
PBEAM, 62, 1, .0203,.703E-4,.251E-2, ,.193E-3, 0.221, +623
+623, YESA, 1.0, , , , +625
+625, , , .03474, , +626
+626', ,'.04025, , .0255
$
PBEAM, 63, 1,.0203,.703E-4,.251E-2, ,.193E-3, 0.221, +633
+633, YESA, 1.0, , , , +635
+635', , ', .03474,', ' ' , '+636
+636', ' .02555, , .0290
$
```

```
PBEAM, \(64,{ }_{1} .0203, .703 E-4, .251 \mathrm{E}-2, \quad .193 \mathrm{E}-3,0.221,+643\)
+643, YESA, \(1.0,1,1,1+645\)
+645, , , . \(03474,,,+646\)
+646', ' . \(.0290^{\prime}\), , . \(0322^{\prime}\)
PBEAM, 65, 1, .0203, .712E-4, .217E-2, , .193E-3, 0.221, +653
+653, YESA, 1.0,
+655,' , , \(.03474, ', ', '+656\)
+656', , '.032'2', , .0376'
PBEAM, \(66,1, .0203, .712 \mathrm{E}-4, .217 \mathrm{E}-2, \quad .193 \mathrm{E}-3,0.244,+663\)
+663, YESA, \(1.0,1,1,1,+665\)
```



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+666', ', .0376', . . \(040025^{\prime}\)
\$PBEAM, 67, 1, .0203, .712E-4, .217E-2, , .193E-3, 0.244, +673
+673 , YESA, \(1.0,1,+675\)
+675', , , .0347 ' \(^{\prime}, ', ',+676\)
+676', ' . .0402́25, , . 0448
\(\$\)
PBEAM, 68, 1, .0203, .712E-4, .217E-2, , .193E-3, 0.244, +683
+683, YESA, 1.0, \(, 1,+685\)
+685',
+686', , '.0550', , . 0402 n \(^{\prime}\)
\(\$\)
PBEAM, 69, \(1, .0203, .729 E-4, .233 E-2,193 E-3,0.2412,+693\)
+693, YESA, 1.0,
+695', , , . \(039{ }^{\prime} 11_{1}^{\prime},{ }^{\prime}, 1+696\)
```



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\$
PBEAM, 70, \(1, .0203, .729 E-4, .233 E-2, ~ .193 E-3,0.2412,+703\)
+703, YESA, \(1.0,1,105\)
+705', , ' 0 , \(03971^{\prime}, ', ~, ~+706\)
+706', ' . .02688, , .0134'
PBEAM, 71, 1,.0203, .729E-4, .237E-2, ,.193E-3, 0.2732, +713
PBEAM, \(71,1, .0203, .729 E-4, .237\)
+713, YESA, 1.0,
```



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+716', ' . .013', , 0.0
\$
PBEAM, \(72,1, .0203, .729 \mathrm{E}-4, .237 \mathrm{E}-2, \quad .193 \mathrm{E}-3,0.2732,+723\)
```



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+726', ' \(\dot{0} .0\),' , -. 0134
\$
PBEAM, 73, 1,.0203, .729E-4, .237E-2, , .193E-3, 0.2732, +733
+733, YESA, \(1.0,1+135\)
+735 , , .04650 , +736
+736', ' '-. \(1134^{.04}\), , \(-0268^{\prime}\)
\(\$\)
PBEAM, 74, 1,.0203, .729E-4, .237E-2, , .193E-3, 0.2732, +743
+743, YESA, \(1.0,1,+745\)
+745', , , \(.04650^{\prime}, ~ ', ~ ' ~+746\)
+746', ', '..0268, , -.0́2 \(47^{\prime}\)
PBEAM, \(75,1, .0203, .729 E-4, .237 \mathrm{E}-2, ~ .193 \mathrm{E}-3,0.2732,+753\)
+753, YESA, 1.0,
+753', YESA, 1.0 .043 ' \(^{\prime}, ~ ', ~ ', ~+756\)
+756, , '-.02477, , -.0249
\(\$\)
\(\$\) PBEAM, 76, \(1, .0203, .729 \mathrm{E}-4, .233 \mathrm{E}-2, \quad .193 \mathrm{E}-3,0.2694,+763\)
+763, YESA, \(1.0,1,1,+765\)
```




```
\$
PBEAM, 77, 1, .0203, .729E-4, .233E-2, , .193E-3, 0.2694, +773
+773, YESÁ, 1.0,
+775', , , , .0435 ' \(^{\prime}, ~, ~, ~, ~ '+776\)
+776, , ' \(-.0255^{\prime}, .,-.0252^{\prime}\)
\$
```



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+785, , , 048554,05%, +786
+786',', -.0252,, , -.0253'
PBEAM, 79, 1,.0203,.729E-4, .233E-2, ,.192E-3, 0.2694, +793
```



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+795,','.'0253, , , -.04254
$
PBEAM, 80, 1, .0203,.729E-4,.233E-2, , .192E-3, 0.2694, +803
+803, YESA, 1.0, , , , ,+805
+805,, ,, .04354,',','+806
+806,', '.0.0254, , -.0255'
$BEAM, 81, 1, .0203, .707E-4, .221E-2, , .192E-3, 0.2646, +813
PBEAM, 81, 1, OESA, 1.0203, .707E-4, .221E-
+815', , ','.0'043'54',''',','816
+816', ', '.0'255, , -.02565
$
PBEAM, 82, 1,.0203,.703E-4, .219E-2, ,.192E-3, 0.2646, +823
+823, YESA, 1.0, , , , +825
+825, , ,'025654354,'',','+826
+826,',':.02565, , -.0258
PBEAM, 83, 1,.0203,.703E-4,.219E-2, ,.192E-3, 0.2646, +833
+833,' YESA, 1.0'03', ,','+835
+836,','.0'255, , -.0268
$
PBEAM, 84, 1, .0203,.703E-4, .219E-2, , .192E-3, 0.2646, +843
+843,'YESA, 1.0'0,'{', ',',+845
+846,: '.0268, , -.0273
$
PBEAM, 85, 1, .0203,.703E-4, .219E-2, ,.192E-3, 0.2646, +853
+853,' YESA, 1.0,
+855, , , , .04354, , , +856
+856,',' -.0273, , -.0279
$
PBEAM, 86, 1, 0203, .690E-4, .177E-2, ,.189E-3, 0.3254, +863
+863,' VESA, 1.0, O, , , , +865
+865, , :, '279.03321'0'8', +866
+866, , -.0'279, , -.0287
PBEAM, 87, 1, .0203,.686E-4,.171E-2, ,.189E-3, 0.3254, +873
+873, YESA, 1.0, O
+875', , ',' .0', 03321,''',','+876
+876', ', '.0188, , , 0537'
PBEAM, 88, 1, .0203, .686E-4,.171E-2, ,.189E-3, 0.3254, +883
+883,' YESA,, 1.0,0, 年, , +885
+885,', ', .03321,''',','+886
+886',',.0537,', -.0349
$
PBEAM, 89, 1, .0203, .686E-4, .171E-2, ,.189E-3, 0.3254, +893
+893,' YESA, 1.0
+895', ,', .03321,','','+896
+896,',' -.0.0349,' , -.072'4'
$
PBEAM, 90, 1, 0203,.612E-4, .158E-2, ,.189E-3, 0.3254, +903
+903,'YESA, 1.0, , , , , +905
+905,',',.05861,',' ','+906
+906,:',.0'7245, -.'1100'
$
PBEAM, 91, 1,.0203, .612E-4, .158E-2, ,.189E-3, 0.3254, +913
+913, YESÁ, i.0, , , , +915
+915,, ,',.05861,''',',+916
+916, ', -.1100, , -.1476'
$
PBEAM, 92, 1,.0203,.612E-4, .158E-2, ,.189E-3, 0.3254, +923
+923, YESA, 1.0,0,, , , +925
+925, , ', .05861,',',''+926
+926,', -.1476,, -. 1038'
```

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$
PBEAM, 93, 1, .0203,.612E-4, .141E-2, ,.189E-3, 0.3254, +933
+933, YESA, 1.0, , , , , , +935
+935', , , '.05723', ',' ' '9336
+936', ', '.1360', -.0939'
$
PBEAM, 94, 1,.0203,.447E-4, .117E-2, ,.189E-3, 0.344, +943
```



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+945', , ' . .05723', ' ',' , '+946
+946', ', '.0939, , -.0357'
$
PBEAM, 95, 1, .0203,.447E-4,.117E-2, ,.189E-3, 0.344, +953
+953,'YESA, 1.0,01`3,', , , +956
+956', ', '.0357, , .0778
$
PBEAM, 96, 1, .0203,.447E-4, .117E-2, ,.189E-3, 0.1698, +963
+963, YESA, 1.0, , , +965
+965, , , '03805,', ' , +966
+966', ', '.2308, , .3350'
$
PBEAM, 97, 1,.0203,.269E-4, .118E-2, , .189E-3, 0.1698, +973
+973, YESA, 1.0, , , , , +975
+975', , ,',.03805,', ' , , +976
+976', ', '.3350', , . 3622'
$
PBEAM, 98, 1, .0203, .269E-4, .118E-2, ,.189E-3, 0.1698, +983
+983, YESA, 1.0, , , , , +985
+985', , ', .03805,', , , +986
+986,', .3622, , .4277'
$BEAM, 99, 1, .0203, .221E-4, .134E-2, , .189E-3, 0.9918, +993
+993, YESA, 1.0, '0, , , , +995
+995', , ,', .03805,',',','+996
+996', , .42'77
$
ENDDATA
```


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Table 1. Measurement locations for the UH-60A blade shake test using coordinate system in figure 9.

| Meas. No. | $\mathbf{X}$ (in.) | $\mathbf{Y}$ (in.) | $\mathbf{Z}$ (in.) |
| :---: | :---: | ---: | :---: |
| 1 | 0.0 | 28.0 | 0.0 |
| 2 | 0.0 | 6.5 | 0.0 |
| 3 | 0.0 | 23.5 | 11.0 |
| 4 | 0.0 | 3.0 | 11.0 |
| 5 | 0.0 | 20.5 | 21.0 |
| 6 | 0.0 | 0.0 | 21.0 |
| 7 | 0.0 | 20.5 | 37.0 |
| 8 | 0.0 | 0.0 | 37.0 |
| 9 | 0.0 | 20.5 | 53.0 |
| 10 | 0.0 | 0.0 | 53.0 |
| 11 | 0.0 | 20.5 | 69.0 |
| 12 | 0.0 | 0.0 | 69.0 |
| 13 | 0.0 | 20.5 | 85.0 |
| 14 | 0.0 | 0.0 | 85.0 |
| 15 | 0.0 | 20.5 | 101.0 |
| 16 | 0.0 | 0.0 | 101.0 |
| 17 | 0.0 | 20.5 | 117.0 |
| 18 | 0.0 | 0.0 | 117.0 |
| 19 | 0.0 | 20.5 | 133.0 |
| 20 | 0.0 | 0.0 | 133.0 |
| 21 | 0.0 | 20.5 | 149.0 |
| 22 | 0.0 | 0.0 | 149.0 |
| 23 | 0.0 | 20.5 | 165.0 |
| 24 | 0.0 | 0.0 | 165.0 |
| 25 | 0.0 | 20.5 | 181.0 |
| 26 | 0.0 | 0.0 | 181.0 |
| 27 | 0.0 | 20.5 | 197.0 |
| 28 | 0.0 | 0.0 | 197.0 |
| 29 | 0.0 | 20.5 | 213.0 |
| 30 | 0.0 | 0.0 | 213.0 |
| 31 | 0.0 | 20.5 | 229.0 |
| 32 | 0.0 | 0.0 | 229.0 |
| 33 | 0.0 | 20.5 | 245.0 |
| 34 | 0.0 | 0.0 | 245.0 |
| 35 | 0.0 | 20.5 | 261.0 |
| 36 | 0.0 | 0.0 | 261.0 |
| 37 | 0.0 | 20.5 | 277.0 |
| 38 | 0.0 | 0.0 | 277.0 |
| 39 | 0.0 | 20.5 | 288.0 |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |
|  |  |  |  |

Table 2. Modal frequencies of the UH-60A blades tested ( Hz ).

| Modes | Prod. <br> Blade 1 | Prod. <br> Blade 2 | Prod. <br> Blade 3 | Prod. <br> Blade 4 | NASTRAN <br> (Production) | Strain <br> Blade | Pressure <br> Blade | NASTRAN <br> (Pressure) |
| :--- | :---: | ---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1st Flap | 4.83 | 4.81 | 4.77 | 4.80 | 4.74 | 4.78 | 4.69 | 4.64 |
| 2nd Flap | 13.01 | 12.77 | 12.77 | 12.74 | 12.94 | 12.74 | 12.46 | 12.71 |
| 3rd Flap | 26.01 | 25.50 | 25.50 | 25.42 | 25.17 | 25.47 | 24.87 | 24.11 |
| 4th Flap | 42.66 | 41.64 | 41.86 | 41.72 | 41.06 | 42.01 | 40.51 | 38.32 |
| 5th Flap | 65.45 | 64.11 | 64.21 | 64.08 | 65.53 | 64.15 | 62.28 | 63.66 |
| 6th Flap | 97.15 | 95.07 | 95.78 | 95.32 | 95.96 | 96.00 | 92.72 | 95.67 |
| 1st Chord | 26.38 | 26.13 | 26.02 | 25.84 | 25.60 | 26.00 | 25.55 | 24.78 |
| 2nd Chord | 70.31 | 69.43 | 69.42 | 69.12 | 69.78 | 69.12 | 67.37 | 67.85 |
| lst Torsion | 46.61 | 45.33 | 45.75 | 45.51 | 44.71 | 45.56 | 44.49 | 44.98 |
| 2nd Torsion | 85.12 | 83.16 | 83.63 | 83.48 | 84.25 | 83.88 | 80.75 | 81.13 |
| weight (lb) | 211.7 | 211.7 | 211.8 | 210.4 | 211.7 | 212.5 | 215.7 | 215.8 |

Table 3. Modal residues of production and instrumented blades.


Table 3. Continued.


Table 3. Continued.

|  | 3rd Flapwise Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq <br> Dam | $\begin{array}{ll} l & 26.01 \\ 6): & 0.22 \end{array}$ | Freq. | $\begin{array}{ll} : & 25.47 \\ b): & 0.17 \end{array}$ | Freq. | $\begin{array}{ll} \text { ): } & 24.87 \\ 6): & 0.18 \end{array}$ |
| Meas No. | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 1.585 | 3.4 | 1.369 | 352.7 | 1.315 | 347.3 |
| 2 | 1.497 | 2.1 | 1.143 | 354.5 | 1.110 | 347.4 |
| 3 | 1.007 | 360.0 | 0.969 | 345.4 | 0.785 | 348.9 |
| 4 | 0.881 | 1.9 | 0.741 | 354.6 | 0.662 | 346.0 |
| 5 | 0.405 | 2.8 | 0.478 | 345.2 | 0.335 | 350.7 |
| 6 | 0.376 | 1.9 | 0.364 | 350.5 | 0.279 | 350.3 |
| 7 | 0.351 | 182.6 | 0.177 | 171.0 | 0.250 | 174.6 |
| 8 | 0.330 | 184.2 | 0.211 | 169.1 | 0.256 | 167.9 |
| 9 | 0.850 | 182.1 | 0.633 | 172.5 | 0.650 | 174.8 |
| 10 | 0.792 | 183.6 | 0.578 | 176.9 | 0.615 | 165.5 |
| 11 | 0.995 | 183.1 | 0.798 | 173.4 | 0.724 | 176.3 |
| 12 | 0.890 | 183.2 | 0.720 | 177.5 | 0.666 | 167.2 |
| 13 | 0.759 | 183.1 | 0.626 | 172.6 | 0.574 | 174.7 |
| 14 | 0.612 | 182.6 | 0.554 | 174.6 | 0.510 | 165.7 |
| 15 | 0.254 | 180.6 | 0.238 | 173.9 | 0.239 | 171.7 |
| 16 | 0.094 | 180.1 | 0.147 | 177.1 | 0.121 | 166.3 |
| 17 | 0.365 | 5.8 | 0.249 | 353.2 | 0.254 | 341.3 |
| 18 | 0.533 | 4.0 | 0.319 | 357.3 | 0.403 | 346.0 |
| 19 | 0.882 | 1.7 | 0.633 | 353.1 | 0.625 | 342.1 |
| 20 | 1.001 | 2.3 | 0.730 | 357.2 | 0.820 | 343.8 |
| 21 | 1.120 | 1.0 | 0.851 | 357.8 | 0.749 | 341.9 |
| 22 | 1.111 | 0.7 | 0.841 | 352.5 | 0.970 | 345.5 |
| 23 | 0.777 | 2.3 | 0.637 | 351.9 | 0.623 | 346.3 |
| 24 | 1.009 | 1.9 | 0.905 | 354.2 | 0.858 | 344.5 |
| 25 | 0.209 | 6.0 | 0.274 | 350.6 | 0.206 | 342.8 |
| 26 | 0.575 | 359.9 | 0.463 | 349.3 | 0.512 | 344.0 |
| 27 | 0.424 | 181.3 | 0.248 | 174.8 | 0.325 | 172.3 |
| 28 | 0.055 | 200.0 | 0.064 | 335.8 | 0.003 | 254.0 |
| 29 | 1.054 | 184.9 | 0.746 | 172.9 | 0.799 | 170.8 |
| 30 | 0.566 | 189.1 | 0.351 | 170.9 | 0.409 | 165.1 |
| 31 | 1.283 | 188.9 | 1.049 | 174.9 | 1.150 | 174.7 |
| 32 | 0.827 | 187.2 | 0.612 | 171.5 | 0.665 | 163.5 |
| 33 | 1.391 | 193.4 | 1.046 | 173.7 | 1.069 | 170.0 |
| 34 | 0.799 | 182.4 | 0.602 | 170.6 | 0.627 | 164.6 |
| 35 | 0.974 | 190.5 | 0.736 | 173.5 | 0.713 | 169.6 |
| 36 | 0.333 | 197.3 | 0.341 | 170.1 | 0.268 | 169.7 |
| 37 | 0.242 | 185.2 | 0.310 | 170.5 | 0.213 | 167.9 |
| 38 | 0.234 | 6.9 | 0.107 | 352.9 | 0.196 | 351.3 |
| 39 | 0.600 | 183.3 | 0.485 | 171.6 | 0.450 | 170.0 |

Table 3. Continued.


Table 3. Continued.

| Meas No. | 5th Flapwise Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. (Hz): <br> Damp. (\%): | $\begin{aligned} & 65.45 \\ & 0.08 \end{aligned}$ | Freq. (Hz): <br> Damp. (\%): |  | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} 62.28 \\ : & 0.14 \end{array}$ |
|  | Ampl. | Phase -Deg | Ampl. |  | Ampl. | Phase-Deg |
| 1 | 2.720 | 359.4 | 2.837 | 353.8 | 2.603 | 359.2 |
| 2 | 3.058 | 359.7 | 3.331 | 353.0 | 2.929 | 2.4 |
| 3 | 0.692 | 359.1 | 0.961 | 351.8 | 0.791 | 54.8 |
| 4 | 1.828 | 359.7 | 1.884 | 351.5 | 1.449 | 360.0 |
| 5 | 1.288 | 179.5 | 0.945 | 172.4 | 1.249 | 186.3 |
| 6 | 0.355 | 0.2 | 0.605 | 350.3 | 0.940 | 179.5 |
| 7 | 3.297 | 179.7 | 3.132 | 170.9 | 3.208 | 180.8 |
| 8 | 1.066 | 179.8 | 1.014 | 172.7 | 0.940 | 179.5 |
| 9 | 3.283 | 180.0 | 3.376 | 170.9 | 3.172 | 180.8 |
| 10 | 0.692 | 180.0 | 0.830 | 173.6 | 0.630 | 180.2 |
| 11 | 1.334 | 179.9 | 1.688 | 171.5 | 1.370 | 180.5 |
| 12 | 0.852 | 0.5 | 0.592 | 353.4 | 0.762 | 359.3 |
| 13 | 0.499 | 0.1 | 0.371 | 353.1 | 0.326 | 358.5 |
| 14 | 2.161 | 0.4 | 2.093 | 352.3 | 2.077 | 356.9 |
| 15 | 0.656 | 359.3 | 0.779 | 353.3 | 0.498 | 357.8 |
| 16 | 2.172 | 0.3 | 2.358 | 351.2 | 2.104 | 356.1 |
| 17 | 0.972 | 179.8 | 0.719 | 172.5 | 0.991 | 176.1 |
| 18 | 0.707 | 359.6 | 0.950 | 351.5 | 0.827 | 354.8 |
| 19 | 2.756 | 180.0 | 2.630 | 173.8 | 2.677 | 176.1 |
| 20 | 1.035 | 180.1 | 0.826 | 173.6 | 1.023 | 175.5 |
| 21 | 3.036 | 179.6 | 3.143 | 173.9 | 2.895 | 177.3 |
| 22 | 1.603 | 179.8 | 1.614 | 172.0 | 1.730 | 176.2 |
| 23 | 1.277 | 178.7 | 1.686 | 171.4 | 1.382 | 176.8 |
| 24 | 0.790 | 180.0 | 1.100 | 170.5 | 0.857 | 176.8 |
| 25 | 1.313 | 2.4 | 0.952 | 350.2 | 1.219 | 358.5 |
| 26 | 0.785 | 1.8 | 0.509 | 352.9 | 0.815 | 357.6 |
| 27 | 3.128 | 0.4 | 3.079 | 351.0 | 2.516 | 358.2 |
| 28 | 1.663 | 359.4 | 1.669 | 349.4 | 1.768 | 356.5 |
| 29 | 2.752 | 359.8 | 2.981 | 351.6 | 2.337 | 357.9 |
| 30 | 1.059 | 358.3 | 1.240 | 354.2 | 1.133 | 355.7 |
| 31 | 0.949 | 357.9 | 1.224 | 349.9 | 0.855 | 355.2 |
| 32 | 0.707 | 180.2 | 0.367 | 172.1 | 0.606 | 176.0 |
| 33 | 0.631 | 180.2 | 0.564 | 176.0 | 0.517 | 178.0 |
| 34 | 2.231 | 179.6 | 1.919 | 170.4 | 2.170 | 175.6 |
| 35 | 0.294 | 179.2 | 0.514 | 174.7 | 0.196 | 176.0 |
| 36 | 2.253 | 181.2 | 2.339 | 170.6 | 1.841 | 175.7 |
| 37 | 1.332 | 0.4 | 0.905 | 350.2 | 1.198 | 357.6 |
| 38 | 0.913 | 179.0 | 1.260 | 170.8 | 0.743 | 174.6 |
| 39 | 0.946 | 180.1 | 1.034 | 170.5 | 0.804 | 178.6 |

Table 3. Continued.

| Meas No. | 6th Flapwise Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} 97.15 \\ 0): & 0.13 \end{array}$ | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} y): & 96.00 \\ b): & 0.16 \end{array}$ |  | $\begin{array}{ll} 92.72 \\ ): & 0.32 \end{array}$ |
|  | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 4.653 | 359.2 | 6.017 | 359.3 | 5.486 | 339.5 |
| 2 | 2.571 | 178.6 | 2.439 | 176.5 | 3.623 | 160.4 |
| 3 | 4.043 | 358.4 | 5.232 | 357.7 | 5.223 | 343.0 |
| 4 | 2.407 | 179.1 | 2.734 | 176.4 | 3.148 | 158.5 |
| 5 | 3.999 | 357.7 | 4.444 | 356.3 | 4.824 | 14.4 |
| 6 | 1.531 | 179.8 | 2.099 | 175.1 | 2.060 | 156.3 |
| 7 | 4.584 | 357.7 | 5.806 | 353.5 | 5.964 | 14.6 |
| 8 | 0.204 | 359.1 | 0.011 | 200.7 | 0.375 | 339.3 |
| 9 | 2.951 | 358.6 | 3.448 | 355.5 | 3.671 | 10.8 |
| 10 | 0.343 | 359.1 | 0.494 | 352.1 | 0.495 | 333.8 |
| 11 | 1.264 | 177.8 | 0.888 | 175.3 | 1.447 | 181.0 |
| 12 | 0.568 | 178.5 | 0.350 | 175.8 | 0.692 | 163.9 |
| 13 | 4.028 | 178.6 | 4.652 | 179.2 | 5.064 | 191.6 |
| 14 | 0.680 | 178.0 | 0.740 | 176.3 | 1.262 | 207.6 |
| 15 | 3.722 | 178.5 | 5.141 | 177.5 | 4.927 | 194.9 |
| 16 | 0.792 | 358.8 | 0.739 | 355.7 | 0.914 | 6.5 |
| 17 | 2.672 | 178.7 | 3.804 | 178.0 | 3.232 | 178.9 |
| 18 | 2.790 | 358.1 | 2.974 | 357.3 | 3.833 | 5.8 |
| 19 | 3.002 | 178.2 | 3.677 | 178.5 | 3.719 | 181.9 |
| 20 | 3.232 | 359.6 | 3.682 | 356.6 | 4.654 | 16.5 |
| 21 | 5.024 | 178.3 | 5.579 | 179.9 | 5.824 | 166.6 |
| 22 | 1.782 | 358.4 | 2.633 | 357.0 | 2.809 | 1.7 |
| 23 | 8.183 | 179.5 | 7.413 | 180.2 | 6.812 | 230.3 |
| 24 | 0.427 | 357.3 | 0.980 | 356.4 | 0.481 | 357.9 |
| 25 | 7.292 | 181.7 | 6.119 | 182.5 | 9.262 | 206.9 |
| 26 | 0.255 | 358.5 | 0.264 | 352.3 | 0.179 | 358.2 |
| 27 | 1.467 | 176.2 | 2.508 | 173.2 | 2.066 | 152.0 |
| 28 | 1.182 | 358.2 | 1.070 | 357.0 | 1.680 | 345.0 |
| 29 | 1.676 | 357.9 | 1.316 | 356.8 | 1.790 | 342.1 |
| 30 | 1.551 | 358.5 | 1.564 | 356.2 | 2.276 | 341.9 |
| 31 | 2.332 | 358.4 | 2.556 | 356.4 | 2.742 | 336.0 |
| 32 | 0.308 | 356.6 | 0.593 | 355.0 | 0.654 | 333.8 |
| 33 | 1.470 | 358.1 | 1.741 | 355.7 | 1.635 | 333.2 |
| 34 | 2.062 | 178.2 | 1.319 | 177.8 | 1.956 | 162.9 |
| 35 | 1.783 | 358.2 | 1.843 | 356.6 | 2.109 | 337.6 |
| 36 | 2.990 | 179.6 | 2.674 | 176.7 | 3.326 | 165.5 |
| 37 | 3.962 | 358.8 | 3.632 | 354.0 | 4.339 | 339.6 |
| 38 | 2.078 | 179.1 | 2.341 | 173.8 | 2.140 | 162.7 |
| 39 | 0.662 | 177.7 | 0.806 | 176.5 | 1.440 | 293.5 |

Table 3. Continued.

| Meas No. | 1st Chordwise Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. (Hz): <br> Damp. (\%): | $\begin{aligned} & 26.38 \\ & : \quad 0.45 \end{aligned}$ | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} : & 26.00 \\ 0): & 0.33 \end{array}$ | Freq. | $\begin{aligned} & 25.55 \\ & : \quad 0.36 \end{aligned}$ |
|  | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 1.754 | 11.1 | 1.777 | 2.5 | 0.097 | 263.2 |
| 2 | 2.025 | 197.8 | 1.708 | 182.6 | 0.150 | 157.3 |
| 3 | 1.469 | 10.4 | 1.553 | 3.0 | 0.100 | 286.9 |
| 4 | 1.473 | 192.0 | 1.150 | 181.3 | 0.127 | 149.8 |
| 5 | 1.096 | 8.3 | 1.225 | 2.3 | 0.120 | 301.9 |
| 6 | 1.235 | 192.9 | 1.150 | 181.3 | 0.072 | 111.6 |
| 7 | 0.714 | 8.5 | 0.758 | 1.1 | 0.112 | 323.7 |
| 8 | 0.681 | 191.9 | 0.745 | 180.1 | 0.043 | 115.1 |
| 9 | 0.281 | 7.2 | 0.311 | 0.7 | 0.030 | 221.1 |
| 10 | 0.289 | 194.5 | 0.305 | 180.3 | 0.032 | 171.3 |
| 11 | 0.221 | 189.8 | 0.095 | 181.3 | 0.007 | 7.0 |
| 12 | 0.122 | 7.6 | 0.074 | 3.6 | 0.029 | 344.0 |
| 13 | 0.458 | 182.5 | 0.432 | 181.7 | 0.020 | 3.8 |
| 14 | 0.515 | 8.0 | 0.458 | 3.3 | 0.054 | 331.2 |
| 15 | 0.745 | 181.5 | 0.736 | 179.6 | 0.034 | 89.7 |
| 16 | 0.781 | 12.6 | 0.755 | 1.7 | 0.066 | 279.2 |
| 17 | 0.974 | 181.6 | 0.984 | 179.8 | 0.073 | 160.0 |
| 18 | 1.042 | 2.3 | 0.950 | 1.2 | 0.086 | 290.7 |
| 19 | 1.118 | 181.7 | 1.204 | 181.8 | 0.100 | 168.6 |
| 20 | 1.149 | 0.9 | 1.117 | 1.1 | 0.093 | 316.0 |
| 21 | 1.128 | 181.2 | 1.194 | 181.0 | 0.124 | 166.0 |
| 22 | 1.447 | 13.8 | 1.219 | 1.2 | 0.092 | 308.3 |
| 23 | 1.122 | 182.1 | 1.187 | 181.6 | 0.132 | 175.0 |
| 24 | 1.452 | 6.9 | 1.168 | 1.6 | 0.105 | 321.0 |
| 25 | 1.022 | 181.9 | 1.069 | 181.7 | 0.132 | 166.1 |
| 26 | 1.289 | 5.0 | 1.078 | 0.9 | 0.151 | 338.9 |
| 27 | 0.812 | 186.7 | 0.896 | 181.4 | 0.130 | 169.1 |
| 28 | 0.942 | 5.0 | 0.966 | 3.3 | 0.115 | 334.6 |
| 29 | 0.620 | 187.9 | 0.660 | 181.8 | 0.101 | 169.4 |
| 30 | 0.600 | 3.2 | 0.674 | 1.9 | 0.094 | 329.6 |
| 31 | 0.385 | 177.8 | 0.382 | 182.1 | 0.084 | 170.3 |
| 32 | 0.312 | 4.1 | 0.360 | 2.2 | 0.066 | 337.4 |
| 33 | 0.010 | 97.1 | 0.013 | 340.4 | 0.055 | 157.2 |
| 34 | 0.043 | 238.2 | 0.019 | 350.0 | 0.009 | 251.3 |
| 35 | 0.387 | 5.4 | 0.341 | 4.7 | 0.036 | 351.9 |
| 36 | 0.456 | 192.5 | 0.344 | 182.3 | 0.030 | 115.2 |
| 37 | 0.716 | 4.9 | 0.738 | 3.4 | 0.086 | 344.9 |
| 38 | 0.747 | 188.1 | 0.698 | 182.0 | 0.086 | 170.6 |
| 39 | 1.014 | 184.3 | 1.023 | 184.7 | 0.210 | 173.1 |

Table 3. Continued.

| Meas No. | 2nd Chordwise Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} \text { ): } & \mathbf{7 0 . 3 1} \\ b): & \mathbf{0 . 3 0} \end{array}$ | Freq. (Hz): <br> Damp. (\%): | $: \quad 69.12$ | Freq. | $\begin{array}{ll} : & 67.37 \\ ) & 0.26 \end{array}$ |
|  | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 4.647 | 182.4 | 4.942 | 180.7 | 4.315 | 180.7 |
| 2 | 4.486 | 2.5 | 4.856 | 357.4 | 4.129 | 2.0 |
| 3 | 3.656 | 182.1 | 3.894 | 180.5 | 2.944 | 180.0 |
| 4 | 3.184 | 1.7 | 3.631 | 359.2 | 2.996 | 1.3 |
| 5 | 2.294 | 183.1 | 2.475 | 180.4 | 1.970 | 179.8 |
| 6 | 2.087 | 1.7 | 2.580 | 356.5 | 1.955 | 2.1 |
| 7 | 0.240 | 182.4 | 0.452 | 181.4 | 0.278 | 182.4 |
| 8 | 0.192 | 0.6 | 0.510 | 357.1 | 0.216 | 0.3 |
| 9 | 1.358 | 2.7 | 1.247 | 0.6 | 1.231 | 1.2 |
| 10 | 1.382 | 183.7 | 1.267 | 179.9 | 1.274 | 180.4 |
| 11 | 2.696 | 2.4 | 2.450 | 359.5 | 2.283 | 0.6 |
| 12 | 2.536 | 182.3 | 2.418 | 179.2 | 2.289 | 180.3 |
| 13 | 3.061 | 2.2 | 3.018 | 0.4 | 2.938 | 0.1 |
| 14 | 3.090 | 182.1 | 3.124 | 179.6 | 2.839 | 181.4 |
| 15 | 2.992 | 359.8 | 3.126 | 0.5 | 2.971 | 1.7 |
| 16 | 3.093 | 181.8 | 3.140 | 180.1 | 2.821 | 180.3 |
| 17 | 2.318 | 358.9 | 2.549 | 359.2 | 2.218 | 359.7 |
| 18 | 2.456 | 179.8 | 2.563 | 179.5 | 2.150 | 179.8 |
| 19 | 1.302 | 359.3 | 1.486 | 359.0 | 1.234 | 0.1 |
| 20 | 1.306 | 179.1 | 1.511 | 179.3 | 1.153 | 179.5 |
| 21 | 0.022 | 216.7 | 0.176 | 359.5 | 0.024 | 184.2 |
| 22 | 0.026 | 146.1 | 0.120 | 182.1 | 0.075 | 1.0 |
| 23 | 1.327 | 180.4 | 1.194 | 180.3 | 1.232 | 181.5 |
| 24 | 1.399 | 359.9 | 1.179 | 359.3 | 1.238 | 0.5 |
| 25 | 2.486 | 179.9 | 2.247 | 180.0 | 2.324 | 181.4 |
| 26 | 2.553 | 359.6 | 2.830 | 344.4 | 2.156 | 0.1 |
| 27 | 3.125 | 179.8 | 2.956 | 180.0 | 2.944 | 180.6 |
| 28 | 3.147 | 359.1 | 3.207 | 0.1 | 2.895 | 359.4 |
| 29 | 3.216 | 179.3 | 3.290 | 179.6 | 3.152 | 179.8 |
| 30 | 3.336 | 359.7 | 3.437 | 359.6 | 3.000 | 359.7 |
| 31 | 2.926 | 179.7 | 3.043 | 179.7 | 2.750 | 179.8 |
| 32 | 3.025 | 359.1 | 3.095 | 0.5 | 2.640 | 359.9 |
| 33 | 2.041 | 181.1 | 2.167 | 179.5 | 1.877 | 179.0 |
| 34 | 1.983 | 351.9 | 2.281 | 359.7 | 1.785 | 1.4 |
| 35 | 0.762 | 176.7 | 0.899 | 178.6 | 0.629 | 178.4 |
| 36 | 0.701 | 349.3 | 0.977 | 359.1 | 0.583 | 358.8 |
| 37 | 0.843 | 2.2 | 0.705 | 1.2 | 0.857 | 359.6 |
| 38 | 0.898 | 182.4 | 0.609 | 180.9 | 0.879 | 179.7 |
| 39 | 2.677 | 181.9 | 2.657 | 182.0 | 2.475 | 179.4 |

Table 3. Continued.

| Meas No. | 1st Torsion Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. Damp | $\begin{array}{ll} 0.61 \\ 0): & 0.13 \end{array}$ | Freq. | $\begin{array}{ll} \text { ): } & 56 \\ 6): & 0.18 \end{array}$ | Freq. | $\begin{array}{ll} : & 44.49 \\ ): & 0.25 \end{array}$ |
|  | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 2.803 | 359.2 | 2.512 | 356.1 | 2.557 | 347.8 |
| 2 | 0.983 | 179.4 | 0.758 | 172.5 | 1.383 | 165.9 |
| 3 | 2.494 | 359.1 | 2.281 | 354.6 | 2.341 | 339.8 |
| 4 | 1.232 | 179.6 | 1.074 | 167.9 | 1.684 | 169.4 |
| 5 | 2.189 | 359.0 | 1.932 | 352.4 | 1.484 | 359.0 |
| 6 | 1.227 | 181.1 | 0.997 | 167.6 | 1.546 | 165.0 |
| 7 | 2.593 | 0.6 | 2.217 | 352.5 | 1.876 | 358.6 |
| 8 | 0.586 | 179.3 | 0.490 | 176.8 | 0.671 | 159.0 |
| 9 | 2.865 | 0.3 | 2.543 | 355.6 | 2.281 | 356.1 |
| 10 | 0.163 | 179.0 | 0.170 | 173.4 | 0.151 | 160.1 |
| 11 | 2.537 | 359.3 | 2.251 | 350.2 | 2.009 | 357.6 |
| 12 | 0.120 | 179.1 | 0.088 | 175.6 | 0.108 | 172.8 |
| 13 | 1.642 | 359.5 | 1.508 | 353.8 | 1.375 | 359.2 |
| 14 | 0.406 | 179.7 | 0.263 | 178.0 | 0.375 | 169.0 |
| 15 | 0.601 | 358.9 | 0.549 | 350.9 | 0.417 | 352.0 |
| 16 | 0.760 | 179.7 | 0.490 | 175.9 | 0.746 | 171.7 |
| 17 | 0.195 | 179.3 | 0.051 | 177.6 | 0.142 | 177.0 |
| 18 | 0.859 | 180.9 | 0.613 | 176.2 | 0.639 | 174.9 |
| 19 | 0.549 | 179.5 | 0.353 | 172.1 | 0.354 | 173.8 |
| 20 | 0.590 | 179.1 | 0.481 | 177.1 | 0.437 | 175.8 |
| 21 | 0.478 | 179.1 | 0.392 | 180.0 | 0.416 | 178.1 |
| 22 | 0.081 | 179.9 | 0.117 | 175.9 | 0.067 | 170.7 |
| 23 | 0.264 | 181.1 | 0.213 | 179.7 | 0.176 | 179.0 |
| 24 | 0.626 | 359.6 | 0.309 | 351.4 | 0.309 | 351.7 |
| 25 | 0.144 | 179.2 | 0.200 | 170.6 | 0.073 | 159.9 |
| 26 | 1.122 | 0.8 | 0.745 | 355.3 | 0.501 | 352.6 |
| 27 | 0.583 | 179.0 | 0.515 | 172.1 | 0.653 | 163.8 |
| 28 | 1.292 | 359.1 | 1.096 | 351.1 | 0.792 | 349.6 |
| 29 | 1.628 | 179.7 | 1.111 | 174.0 | 1.607 | 168.6 |
| 30 | 1.056 | 358.2 | 0.855 | 348.4 | 0.575 | 349.9 |
| 31 | 2.604 | 180.7 | 1.903 | 174.9 | 2.535 | 177.0 |
| 32 | 0.634 | 358.5 | 0.570 | 350.7 | 0.310 | 349.1 |
| 33 | 3.263 | 179.6 | 2.470 | 178.0 | 3.387 | 180.1 |
| 34 | 0.286 | 358.5 | 0.351 | 353.7 | 0.164 | 350.7 |
| 35 | 3.269 | 179.3 | 2.517 | 179.6 | 3.440 | 177.4 |
| 36 | 0.469 | 358.7 | 0.496 | 350.2 | 0.453 | 355.3 |
| 37 | 2.531 | 179.1 | 2.388 | 171.4 | 2.884 | 185.9 |
| 38 | 0.848 | 359.5 | 0.734 | 357.2 | 0.915 | 356.6 |
| 39 | 0.343 | 180.1 | 0.230 | 174.2 | 0.371 | 176.0 |

Table 3. Concluded.

| Meas No. | 2nd Torsion Mode |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Production Blade |  | Strain Gage Blade |  | Pressure Blade |  |
|  | Freq. (Hz): <br> Damp. (\%): | $\begin{array}{ll} 85.12 \\ ): & 0.19 \end{array}$ | Freq. | $\begin{array}{ll} \text { ): } & 83.88 \\ \text { ): } & 0.13 \end{array}$ | Freq. Damp | $\begin{array}{ll} : 80.75 \\ ): & 0.20 \end{array}$ |
|  | Ampl. | Phase -Deg | Ampl. | Phase -Deg | Ampl. | Phase-Deg |
| 1 | 8.729 | 179.4 | 8.849 | 188.9 | 7.002 | 206.2 |
| 2 | 2.332 | 180.9 | 2.452 | 176.7 | 1.735 | 195.6 |
| 3 | 4.577 | 180.3 | 5.143 | 179.4 | 4.067 | 190.2 |
| 4 | 0.113 | 356.4 | 0.249 | 178.8 | 0.464 | 14.6 |
| 5 | 1.175 | 179.0 | 1.459 | 175.7 | 0.804 | 196.4 |
| 6 | 1.517 | 0.8 | 1.130 | 356.7 | 1.584 | 12.8 |
| 7 | 0.797 | 2.0 | 0.806 | 354.6 | 0.687 | 3.6 |
| 8 | 1.239 | 359.0 | 1.158 | 357.3 | 1.057 | 12.4 |
| 9 | 0.384 | 2.3 | 0.528 | 356.0 | 0.378 | 5.7 |
| 10 | 0.641 | 179.8 | 0.254 | 176.2 | 0.731 | 191.5 |
| 11 | 0.201 | 2.8 | 0.069 | 357.6 | 0.338 | 7.4 |
| 12 | 2.075 | 180.3 | 1.681 | 176.2 | 2.084 | 188.7 |
| 13 | 2.019 | 2.3 | 1.349 | 357.8 | 1.809 | 9.4 |
| 14 | 1.944 | 180.5 | 1.851 | 177.7 | 1.662 | 185.2 |
| 15 | 4.764 | 0.9 | 3.600 | 357.5 | 3.836 | 14.6 |
| 16 | 0.711 | 180.1 | 0.685 | 176.7 | 0.614 | 182.4 |
| 17 | 6.541 | 0.6 | 5.125 | 357.4 | 4.638 | 3.3 |
| 18 | 0.048 | 4.7 | 0.126 | 358.3 | 0.002 | 39.7 |
| 19 | 5.482 | 0.6 | 5.058 | 0.4 | 3.826 | 2.6 |
| 20 | 0.701 | 180.0 | 0.307 | 179.0 | 0.474 | 180.7 |
| 21 | 2.791 | 0.9 | 2.696 | 358.0 | 2.419 | 7.6 |
| 22 | 2.382 | 181.3 | 1.538 | 179.2 | 1.352 | 183.6 |
| 23 | 0.795 | 358.1 | 0.719 | 356.7 | 0.957 | 6.9 |
| 24 | 2.896 | 181.2 | 2.285 | 178.5 | 1.526 | 182.1 |
| 25 | 1.072 | 2.8 | 0.705 | 356.7 | 0.675 | 0.2 |
| 26 | 1.701 | 180.5 | 1.670 | 175.8 | 0.858 | 179.5 |
| 27 | 2.335 | 359.7 | 2.071 | 354.7 | 2.179 | 9.6 |
| 28 | 0.398 | 3.2 | 0.045 | 30.8 | 0.286 | 6.1 |
| 29 | 2.475 | 359.9 | 2.345 | 356.0 | 2.053 | 8.6 |
| 30 | 1.677 | 359.1 | 1.389 | 358.4 | 1.125 | 4.6 |
| 31 | 0.169 | 199.2 | 0.460 | 351.5 | 0.202 | 193.0 |
| 32 | 1.194 | 358.5 | 1.216 | 358.1 | 0.822 | 1.7 |
| 33 | 3.758 | 181.0 | 2.735 | 179.0 | 3.211 | 188.6 |
| 34 | 0.037 | 336.8 | 0.128 | 350.0 | 0.017 | 336.3 |
| 35 | 4.789 | 180.4 | 4.024 | 178.1 | 4.088 | 193.4 |
| 36 | 0.155 | 180.6 | 0.295 | 176.1 | 0.118 | 182.8 |
| 37 | 3.776 | 180.6 | 3.799 | 176.2 | 3.485 | 190.0 |
| 38 | 0.880 | 3.9 | 0.517 | 356.9 | 0.823 | 5.9 |
| 39 | 0.725 | 182.4 | 0.616 | 177.5 | 0.662 | 189.5 |



Figure 1. The UH-60A Black Hawk.


Figure 2. The UH-60A pressure-instrumented blade top and bottom transducer layout.

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Figure 3. Pressure-instrumented blade tip showing pressure transducers (smaller holes are pressure transducers).


Figure 4. The CBEAM element used to construct the NASTRAN models .


Figure 5. Chordwise stiffness versus blade radial position for production and pressure-instrumented blades (courtesy of Sikorsky Aircraft).


Figure 6. Shake test set-up showing a production blade hanging from bungee chords.

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Figure 7. Blade support and shaker used in the shake test.


Figure 8. Schematic of the test set-up.


Figure 9. Measurement locations on the UH-60A blades.


Figure 11. Modal frequencies measured for the production (average of four blades), strain-gage, and pressureinstrumented blades.


Figure 12. Calculated versus measured production blade frequencies.


Figure 13. NASTRAN prediction of pressure-instrumented blade frequencies.

| N/Sn |  |  |
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| 16. Abstract <br> The dynamic characteri rotor blades were measured models. The blades tested i part of NASA's Airloads Fl The dynamic similarity of Therefore, a nonrotating bla measure blade similarities. cies of instrumented and prod els. This type of modal test instrumented blade flight te | cs of instrumented and pro and the results were validated luded pressure and strain-g t Research Phase of the M blades was required for ac modal analysis was perfo e results showed small diff uction blades and a close c and analysis is recommen ing. | -60A Black Hawk main STRAN finite element nented blades, which are nology Rotor Program. collection in this program. e first 10 free-free modes to tween the modal frequenwith the NASTRAN modandard procedure for future |
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