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CENTER FOR SPACE STRUCTURES AND CONTROLS

**IMPLEMENTATION OF A  
PARTITIONED ALGORITHM  
FOR SIMULATION OF LARGE  
CSI PROBLEMS**

by

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March 1991

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**Implementation of a Partitioned Algorithm  
for Simulation of Large CSI Problems**

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## Summary

This report summarizes research work on the implementation of a partitioned numerical algorithm for determining the dynamic response of coupled structure/controller/estimator finite-dimensional systems. The partitioned approach leads to a set of coupled first and second-order linear differential equations which are numerically integrated with extrapolation and implicit step methods. The present software implementation, ACSIS, utilizes parallel processing techniques at various levels to optimize performance on a shared-memory concurrent/vector processing system. The current work also generalizes the form of state estimation, whereby the Kalman filtering method is recast in a second-order differential equation equivalent to and possessing the same computational advantages of the structural equations. As part of the present implementation effort, a general procedure for the design of controller and filter gains is also implemented, which utilizes the vibration characteristics of the structure to be solved. Example problems are presented which demonstrate the versatility of the code and computational efficiency of the parallel methods is examined through runtime results for these problems. A user's guide to the ACSIS program, including descriptions of input formats for the structural finite element model data and control system definition, can be found in Appendix A. The procedures and algorithm scripts related to gain design using PRO-Matlab are included in Appendix B. In Appendix C, a stability analysis for the partitioned algorithm is presented which extends previous analysis to include observer dynamics, leading to a clearly definable stability limit. The source code for the parallel implementation of ACSIS is listed in Appendix D.

## 1.0 Introduction

The present work on the implementation of a partitioned transient analysis algorithm for the simulation of linear Control-Structure Interaction (CSI) problems has concentrated on four major areas. The initial software implementation emphasized the user-friendly aspect and a structural dynamics-oriented interface for experienced practitioners of finite element analysis programs. Another reason for a new implementation of the algorithm was that the initial architecture of the CS3 software testbed developed by Belvin and Park [1-2] had little provision for effective parallelization. CS3 also included extensive links to optimization and optimal control algorithms which were not central to the current work and proved to be a further hinderance. This new software implementation was designated ACSIS, for Accelerated Control-Structure Interaction Simulation, and led to significant improvements in speed for particular problems on conventional serial processors due to simpler and more economical storage of primary variables. ACSIS is also versatile in its usage of a general Timoshenko beam element with pin release capability and shear correction factor adjustment, and control system definition via a single data file. A preliminary Users Guide was developed for ACSIS, and the input formats were made compatible with pre/postprocessing software developed for Sun and Silicon Graphics computers so that

future versions using parallel techniques via element mesh domain decompositions can rely on the same X-Window-based I/O utilities. Table 1 documents runtime comparisons of ACSIS and CS3 on a sample 48 DOF problem simulated on a Sun 3/260 workstation using a floating point accelerator.

## 2.0 State Estimation via Second-Order Kalman Filtering

One restriction of the CS3 software testbed was the form of dynamic observer equations used in the partitioned algorithm as developed in [2]. However, Belvin and Park [3] showed that a general Kalman filtering type of state estimation was not only possible in the partitioned solution, but could be implemented at a very small additional cost in computations by a slight modification in the way the dynamical equations of the plant are cast into first-order form for filter gain design. The methods employed in [3] have been successfully implemented into ACSIS, thereby enhancing the code's ability to handle a wide variety of state estimation schemes. This is important as the application of optimal control techniques to the state estimator problem leads to this more general form, and as the restrictions of the observer used in CS3 typically meant either discarding part of the observer gain parameters, resulting in a loss in system performance.

ACIS retains the option of using the more restricted observer form, as well as simple full state feedback (no dynamic compensation). It is clear from examining the respective equations, that for structures with stiffness-proportional damping, the only additional calculations required for the Kalman filter is the multiplication of the  $L_1$  gain matrix by the predicted state estimation error vector  $\gamma$  (see equations 25 of [3]). This is not a significant portion of the computations required at each integration step, as the dimension of  $\gamma$  is small (number of sensors). By far the major costs are for computing an internal force of the form  $Kq$ , and backsolving the factored integration matrix for the estimated displacement states. This is verified numerically in the ACSIS results of Table 1 (using a restricted form of observer) and version (A1) of Table 2. The model used in Table 1 is roughly comparable to the 54 dof truss in Table 2, and both show the additional simulation time due to state estimation is roughly the same as an additional transient analysis. Finally, the Kalman filtering equations do not lead to any additional complications in parallel implementation of the overall algorithm as compared to the more restricted second-order observer developed for CS3.

### 3.0 Parallel Implementation of ACSIS

A primary emphasis in our work dealt with the optimization of the software implementation on a concurrent processing system. The platform chosen (primarily due to availability, initially at CU and later at NASA LaRC) was the Alliant FX/8 shared memory multiprocessor system with 8 parallel processing units and vectorization capabilities. Versions of the software have been ported to the Alliant and compiled using the Concentrix FX/Fortran optimizing compiler, which has available options for automatic vectorization and concurrency of standard, problem-independent parallel computations.

The partitioned CSI algorithm has three primary levels of parallelism in its numerics which can be exploited. At the highest level is the integration of the second-order dynamical plant (structure) and filter (estimator) equations, which as designed are of roughly equivalent size. Through the algorithm, these systems are effectively decoupled and independent at each discrete time step, and thus may be handled in parallel by invoking a compiler directive in the main program, which calls the respective subroutines simultaneously and handles re-synchronization of the execution upon their return.

At a lower level of the algorithm, the structure and state estimator equations exhibit symmetric, second-order forms typical of linear structural dynamics problems. It is well known that computations related to the formation of the internal force vector,  $Kq$ , can be re-implemented at an element level [4], which, through decomposition of the element mesh [5], can be handled in parallel within exclusive subdomains. This technique becomes particularly attractive for larger problems on more massively parallel systems; in the present work, the element-by-element (EBE) technique was not as effective as other types of parallelism. It should be noted here that the partitioned algorithm employs implicit integration methods, leading to systems of algebraic equations which are factored and solved using direct, rather than iterative, numerical methods. Therefore, formulation of the internal force vector is needed only in the formation of the known (right hand side) vector for integration of the plant and filter equations. The alternative to EBE computations is multiplication of the relevant displacement vector with the global stiffness matrix stored in profile (skyline) form. To "simulate" the advantages of local memory typical in large-scale parallel processing systems, the computed element stiffnesses were saved in shared memory for the EBE calculations, thus avoiding the need to recompute this data at each integration step. In addition, a low-overhead automatic element domain decomposition was provided for the parallel EBE method.

The final and lowest level of parallelism is obviously that of the basic matrix computations such as addition, multiplication, etc. These numerical operations are inherent in nearly all areas of the software implementation, including problem preprocessing. With the very capable optimizing compiler available on the Alliant system, this parallelism was exploited through vectorization and concurrency of the nominal source code using the FX/Fortran compiler run-time options `-O -DAS -alt`. The performance of the resultant executable code was examined using the Alliant's profiling capabilities, and changes to the nominal

source code, remaining compliant to F77, to maximize the identifiable concurrency and thus enhance the resultant speed. This was particularly useful in the profile matrix/vector multiply operation, whose speed is critical to the overall program performance.

#### 4.0 Problem Descriptions

Three structural dynamics problems were developed for code testing at various levels of complexity. All three problems have the following common features: simulations consisted of 1000 integration steps and employed a stiffness-proportional damping. The damping was not needed for algorithm stability, but to ensure consistency between the examples and because the existence of damping in the plant equations has a strong influence on program speed. The Kalman filter models were of second-order form [2] and equivalent in size to their respective plant models. For controlled simulations, the control system began operating after 100 integration steps, and all gain matrices were full (i.e. all model states influence all actuators and are influenced by all sensors). Additional specific information for the problems follow.

##### A. Axial Vibration of Elastic Bar (Spring Model)

# Nodes:	3 free, 1 fixed
# Elements:	3
# Degrees of freedom:	3
# Actuators:	1
# Sensors:	1
Disturbance:	Initial displacement

##### B. Planar Vibration of Space Truss (Truss Model)

# Nodes:	18 free
# Elements:	33
# Degrees of freedom:	54
# Actuators:	4
# Sensors:	6
Disturbance:	Bang-bang type sinusoidal applied force

##### C. General 3D Vibration of EPS Satellite Reboost (EPS7 Model)

# Nodes:	97 free
# Elements:	256
# Degrees of freedom:	582
# Actuators:	18
# Sensors:	18
Disturbance:	Bang-bang type square wave applied force

As can be seen, the problem sizes are roughly three different orders of magnitude, with corresponding increases in the sizes of the control systems. Appendix B includes routines using Matlab for the design of control and filter gains which were used for the control system design of all three example problems. An illustration of the EPS model is shown in Figure 1.

## 5.0 Performance Assessment

Table 2 compares CPU runtimes on the Alliant computer (using the UNIX "time" command) for distinct versions of the software. Version (A1) is the nominal F77 program code compiled without any performance-enhancing options, while version (A2) invokes automatic vectorization and concurrency of low-level, problem-independent computations such as vector addition and inner products. The performance improvements are significant, especially for the large EPS7 model, where the speed-up factor is 35-37.

Version (A3) also uses the compiler options from (A2), but in addition has a compiler directive added to the main program which allows the plant and filter integration subroutines to be called in parallel. This does not affect the transient response results as that analysis option bypasses the altered code, but for controlled response there is some effect on performance. If filtering is used, which results in a significant increase in computation, the directive can lead to some increased speed as can be seen for the spring and truss problems. There can also, however, be a reduction in performance as compared to (A2) if the finite amount of processors and vector units are used in a less efficient way. This appears to be the case for the large EPS problem, where the "overhead" introduced by the directive, and its effect on processor assignment, is greater than the improvement generated by the manually-invoked parallel construct.

Table 3 shows CPU run times for ACSIS using E-B-E computations, which, as mentioned previously did not lead to better performance on the example problems using the Alliant system. This appears to be due to the lack of effective vectorization of the individual element computations when forming the internal force via the EBE method. To determine whether the EBE method effectively lead slower speeds through increased numbers of computations, version (A2) (see above) was altered by removing compiler optimization of the profile matrix/vector multiply routine (the alternate method to EBE). The resultant runtimes matched almost exactly with those of version (A4), leading to the conclusion that both methods require roughly the same amount of computations, but differ in how they can be optimized on the Alliant system. The matrix/vector multiply operation, in this environment, can exploit both vectorization and concurrency through the compiler's performance options; this can be examined in the compiler output. The EBE computations, at the element level, do not vectorize because the parts of the global displacement and force vectors being operated on per element are not contiguous. In version (A5), the elements within each subdomain of the mesh are computed in parallel, leading to some performance

Overall, versions (A2) and (A3) provide the best code performance for the hardware available. Parallelizing the observer and structure (A3) leads to mixed results; improvement for the small spring and truss problems, but not for the large EPS model. Element-by-element computations do not improve code performance over compiler optimization via vectorization and concurrency for this platform. Reimplementation of the algorithm lead to a 5:1 improvement over the CS3 testbed software on a serial computer (Table 1). Further optimization of ACSIS on the Alliant FX/8 lead to an additional 30:1 improvement in runtimes for large-order systems such as the 582 dof EPS model. Time history responses of selected variables for the example problems are shown in Figures 2 through 10. For Figures 8 through 10, the  $u_x, u_y, u_z$  displacements plotted are located at node 45, which is located at the vertex of the large antenna of the EPS model (see Figure 1).

## 6.0 Conclusions

The present work has demonstrated the efficiency of a streamlined simulation code for the analysis of large-order CSI systems and the viability of the continuous second-order Kalman filtering equations for state estimation. These methods show the versatility of the partitioned CSI integration algorithm and the promise of its application to real-time simulation. It is evident, however, that the use of element-by-element computational techniques requires the development of innovative algorithms for effective implementation on massively parallel processing systems. Future work in this area will include integrating algorithms for on-line system identification and applying these capabilities to the problem of real-time control.

## Acknowledgements

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Simulation	CS3	ACSIS
Transient	439.2	98.8
Full State Feedback	688.2	181.5
FSFB with Observer	1156.7	282.3

Table 1: Comparison of Runtime Speeds for CS3 and ACSIS  
on a Sun 3/260 System

Model	Problem Type	(A1)	(A2)	(A3)
		Nominal Code	Compiler Optimized	Parallel Observer
3 DOF Spring	Transient	6.6	2.1	2.1
	FSFB	8.0	3.3	3.3
	K. Filter	12.3	3.5	3.3
54 DOF Truss	Transient	78.2	5.7	5.6
	FSFB	97.1	9.4	10.2
	K.Filter	170.7	13.0	10.7
582 DOF EPS7	Transient	3506.	98.6	100.3
	FSFB	7040.	190.2	294.5
	K. Filter	n/a	284.2	312.5

Table 2: CPU Results for Versions of ACSIS

Model	Problem Type	(A4)	(A5)	(A6)
		E-B-E Computation	Parallel E-B-E	Parallel Obs. & EBE
3 DOF Spring	Transient	3.8	3.3	3.3
	FSFB	4.9	4.4	4.9
	K. Filter	6.6	5.6	5.0
54 DOF Truss	Transient	31.7	13.0	13.0
	FSFB	35.5	16.9	35.6
	K.Filter	62.6	27.3	36.2
582 DOF EPS7	Transient	391.7	153.9	n/a
	FSFB	485.9	245.9	n/a
	K. Filter	n/a	n/a	n/a

Table 3: CPU Results for ACSIS with EBE Computations

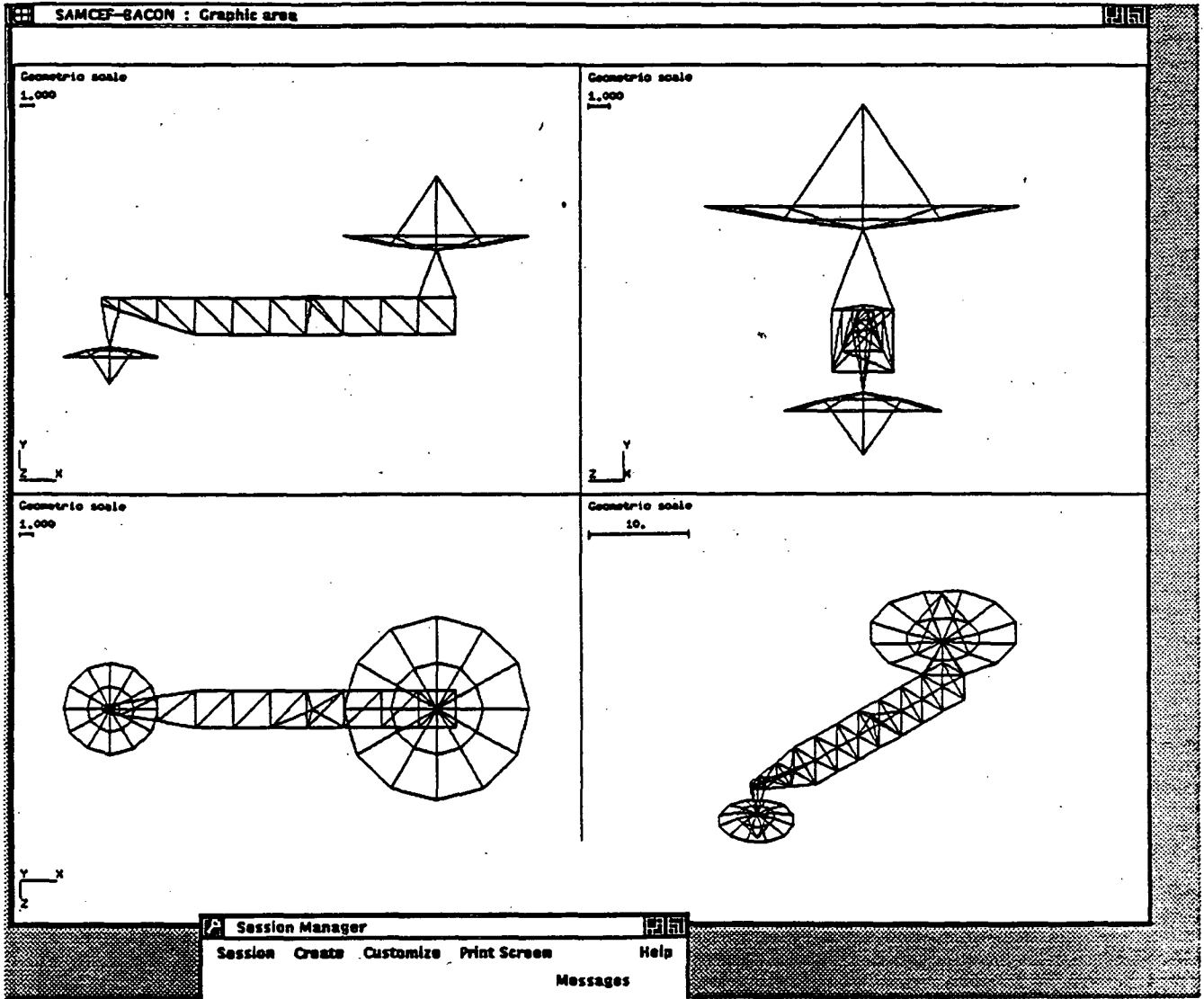
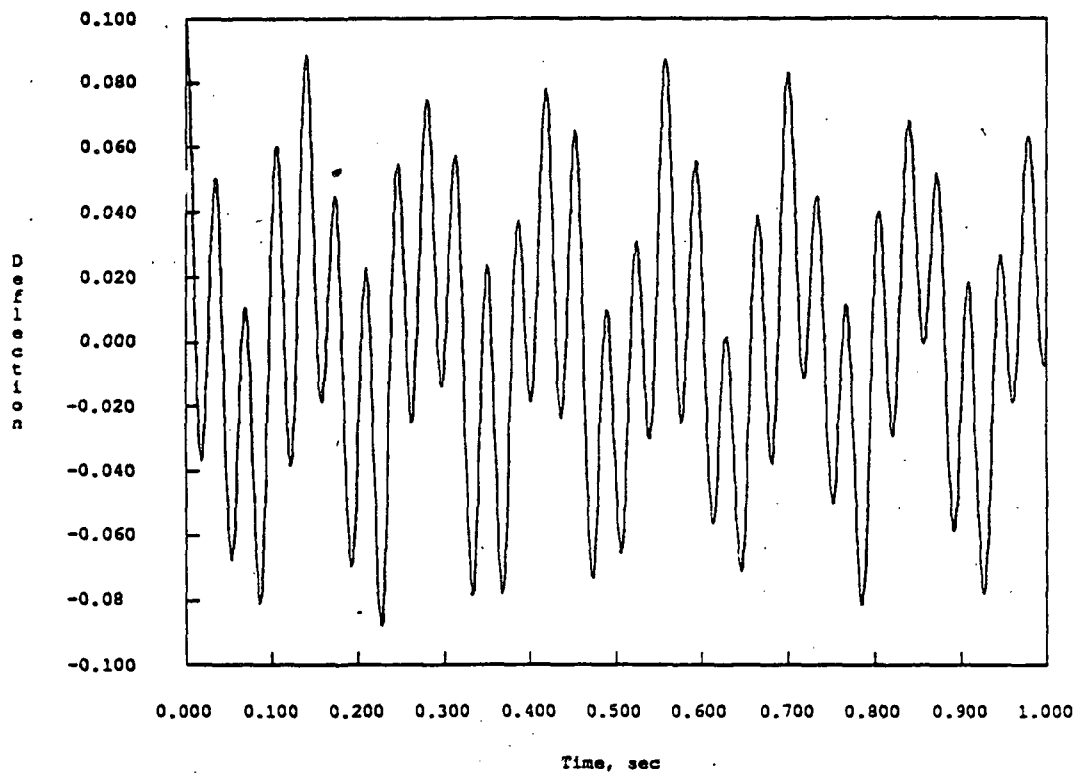


Figure 1: EPS Finite Element Model

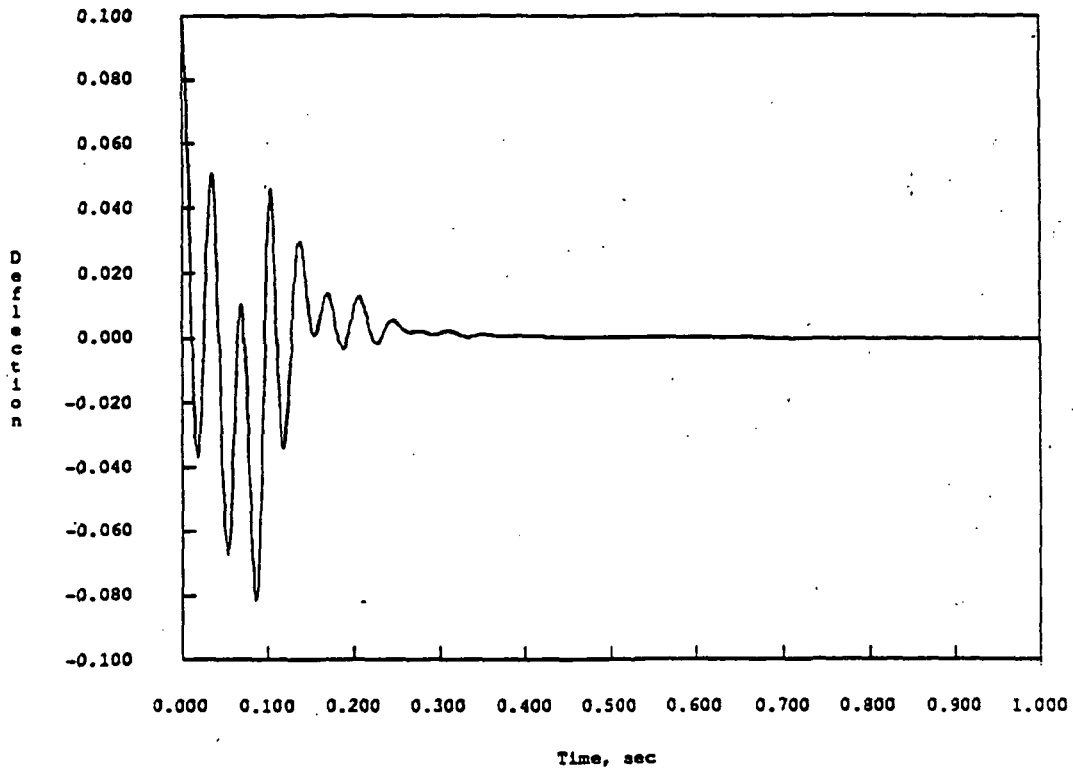
Spring Model: Open Loop Transient Response



— Node 3, ux

Figure 2: Spring Transient Response

### Spring Model: Full State Feedback Response



— Node 3, ux

Figure 3: Spring FSFB Response

Spring Model: Controlled Response w/Kalman Filter

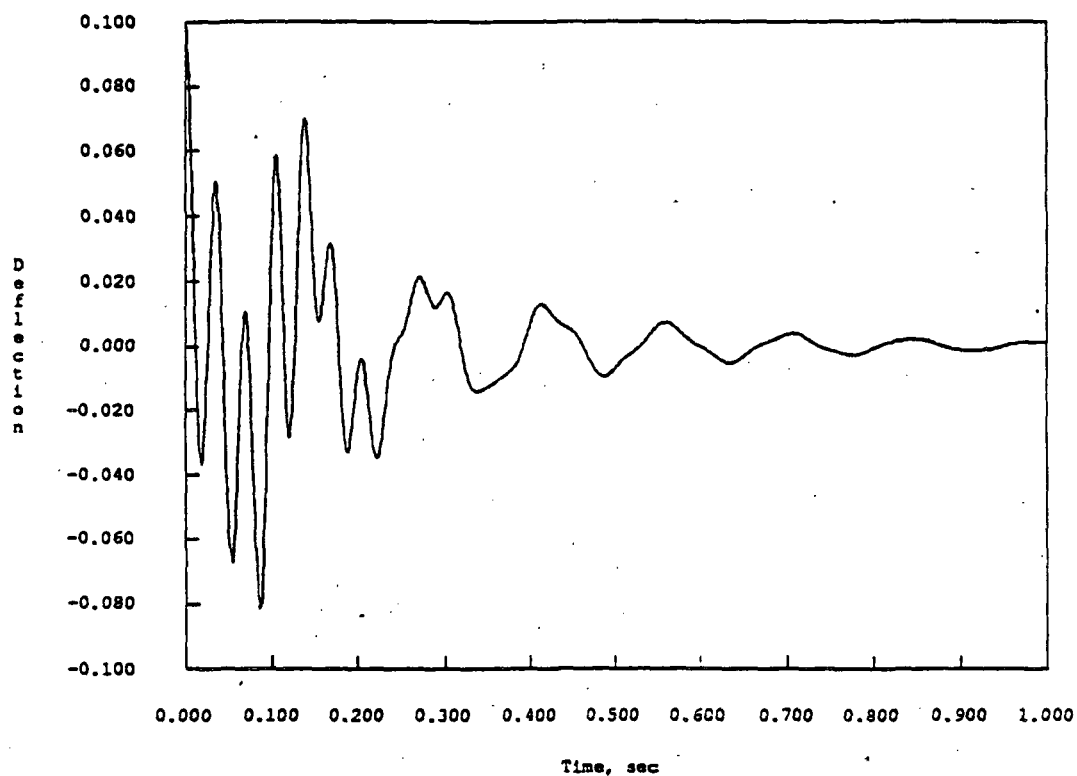


Figure 4: Spring Response w/Filter

### Truss Model: Open Loop Transient Response

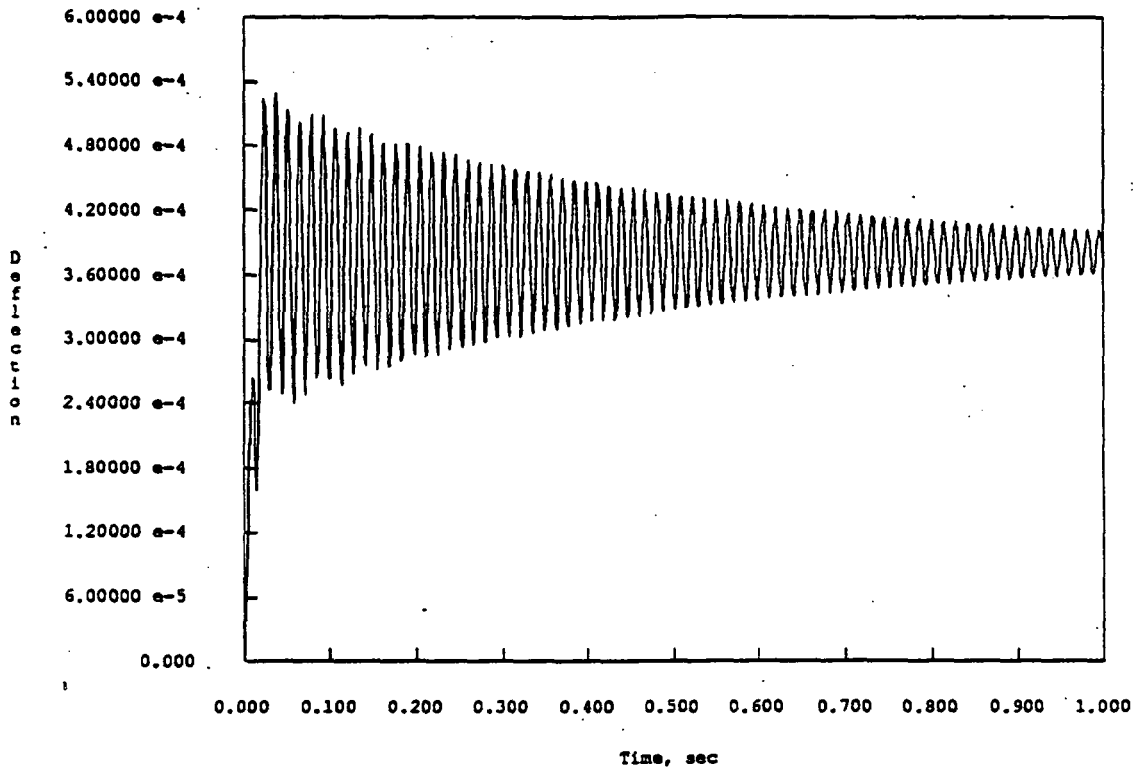
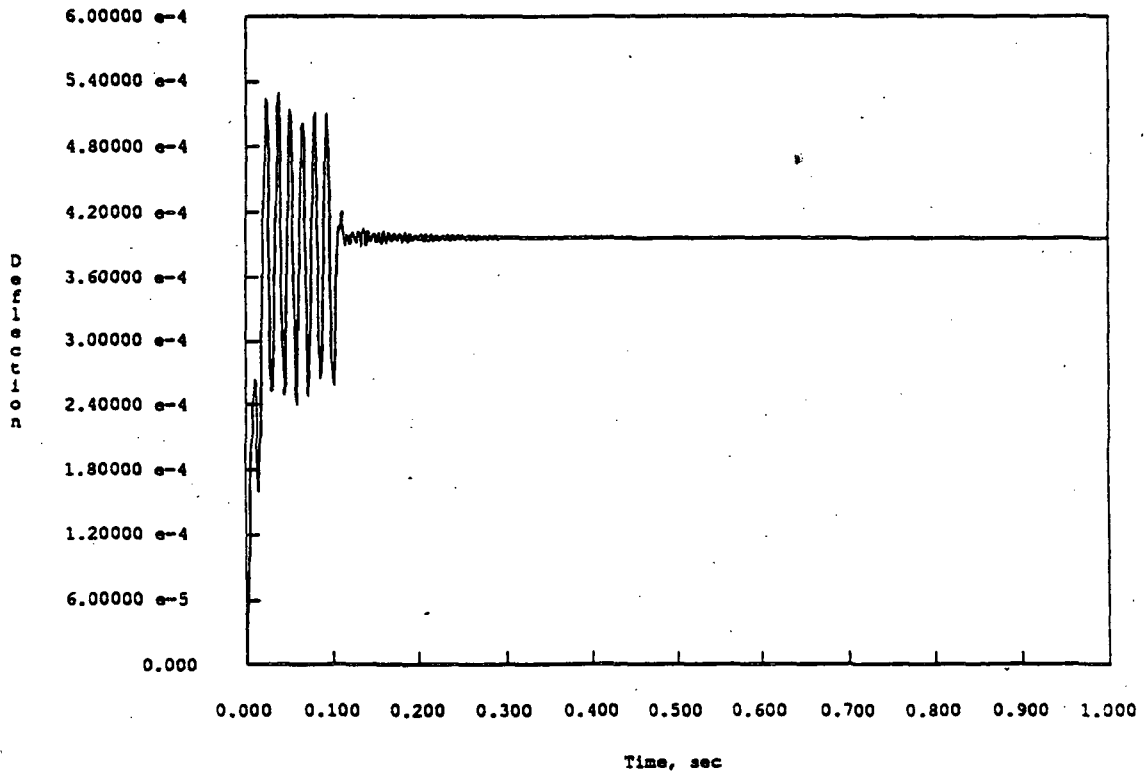


Figure 5: Truss Transient Response

Truss Model: Full State Feedback Response

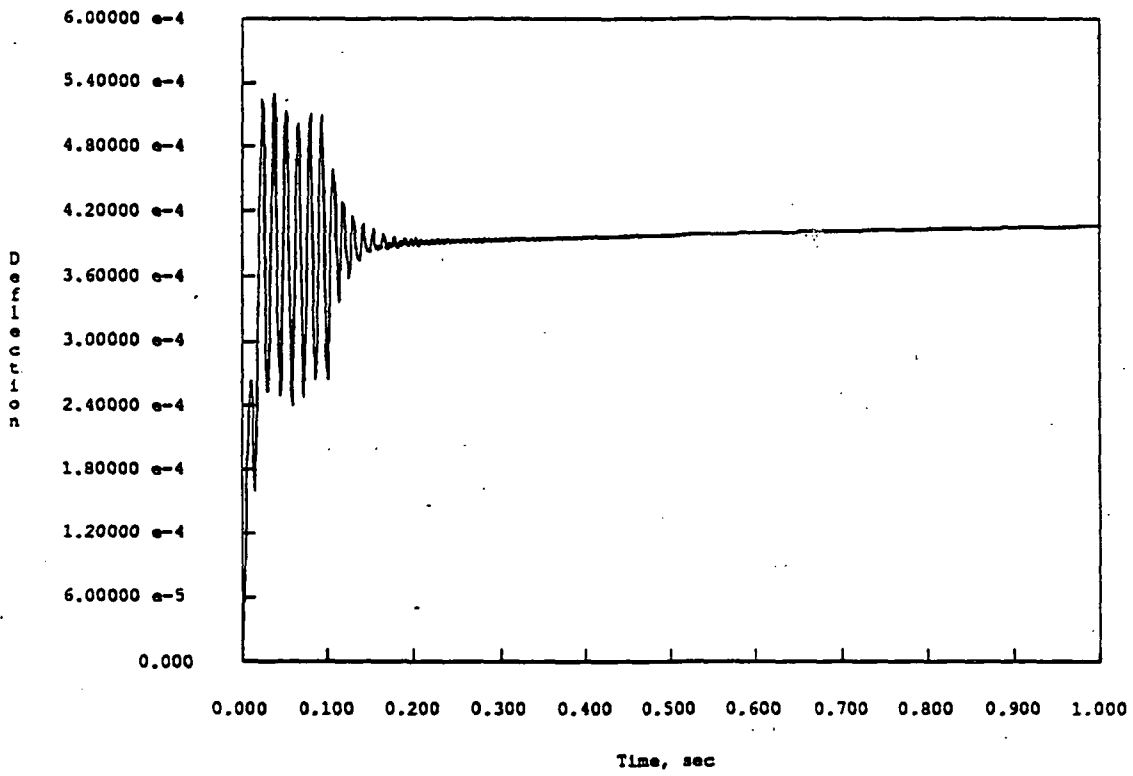


— Node 9, uy

Figure 6: Truss FSFB Response



Truss Model: Controlled Response w/Kalman Filter



— Node 9,  $u_y$

Figure 7: Truss Response w/Filter

### EPS7 Model: Open Loop Transient Response

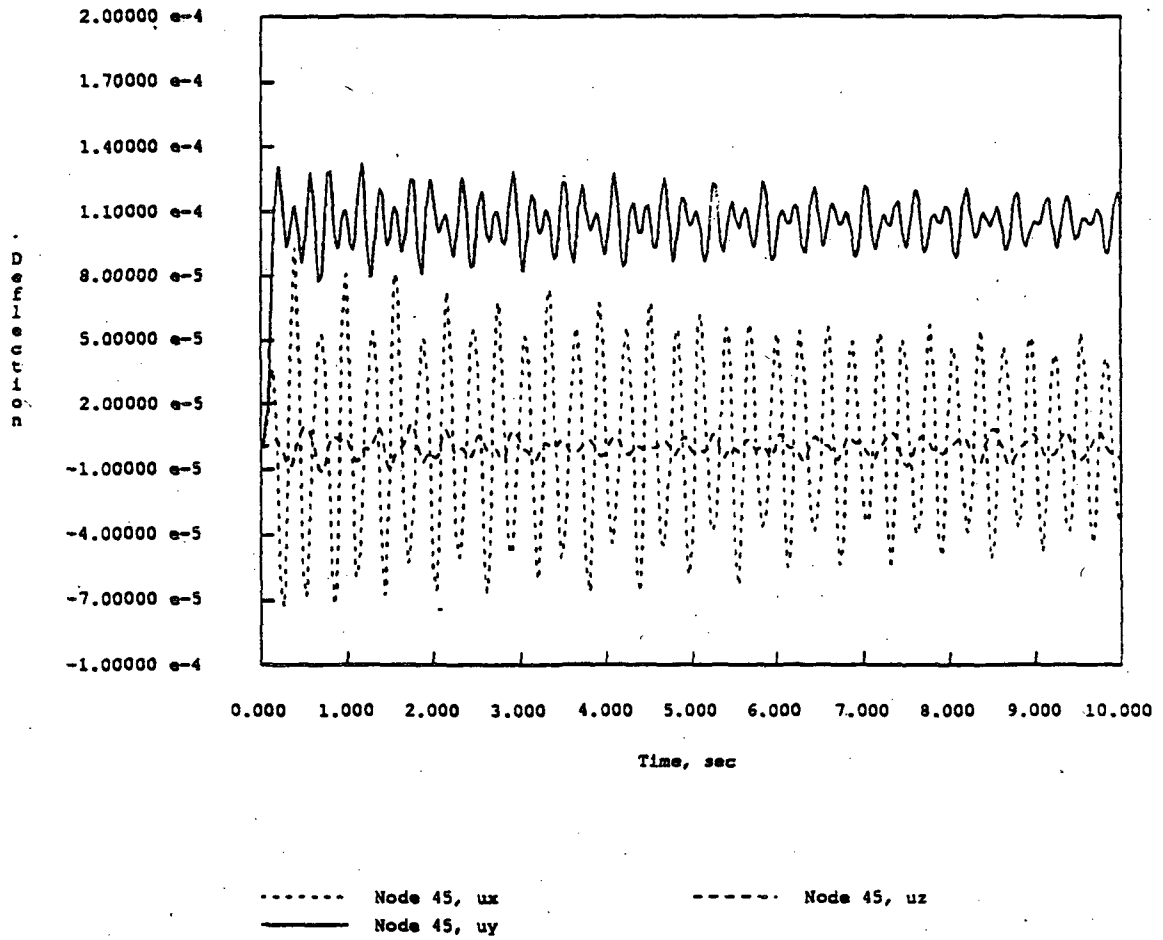
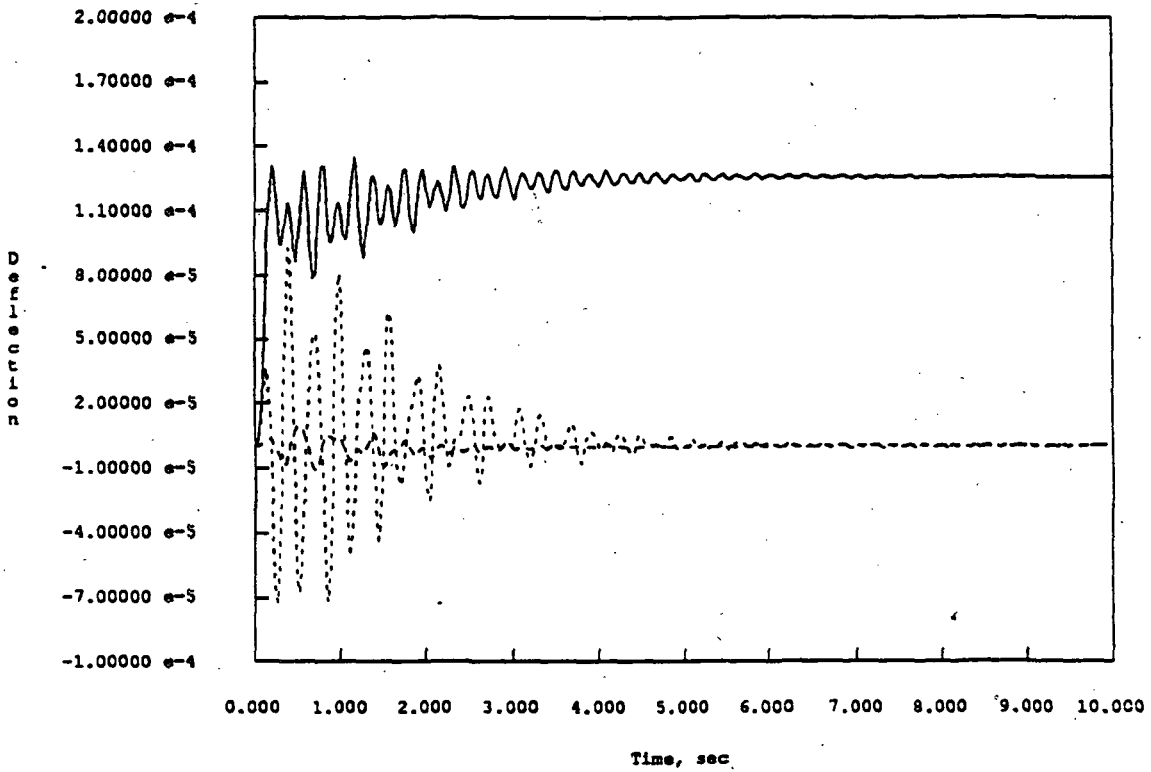


Figure 8: EPS Transient Response

### EPS7 Model: Full State Feedback Response



----- Node 45, ux                      - - - - - Node 45, uz  
\_\_\_\_\_ Node 45, uy

Figure 9: EPS FSFB Response

EPS7 Model: Controlled Response w/Kalman Filter

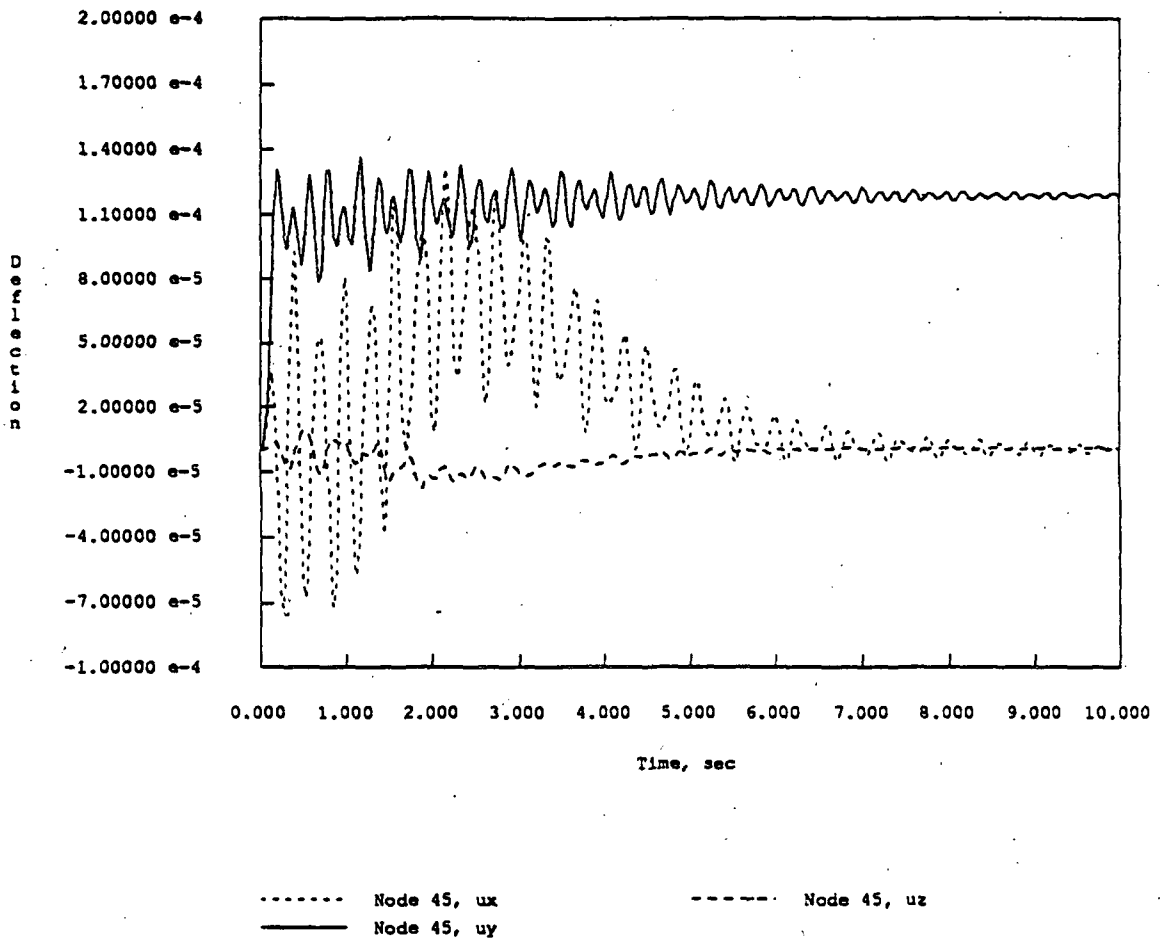


Figure 10: EPS Response w/Filter

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**APPENDIX A**

**ACSIS User's Manual**

**Accelerated  
Control  
Structure  
Interaction  
Simulation**

**Introduction**

ACSIS is an analysis program for full-order simulation of control-structure interaction (CSI) problems. The CSI simulation is carried out using a partitioned analysis procedure which treats the structure (or plant), the observer, and the controller/observer interaction terms as separate entities. This procedure allows ACSIS to maintain relatively small, sparse matrix equations when compared to the process of assembling the computational elements into a single set of equations of motion and solving simultaneously. Although this software can also carry out modal analysis, the transient response and CSI simulation are done in real space using the entire finite element model. (For more information, see Ref. 2 of report, p. 14)

The ACSIS program is run using two previously prepared data input files and interactive input of run options. The two data input files are the finite element input file and the controller definition file.

## Interactive Options

The run options are as follows, note that no defaults exist, data must be input each time it is requested for each item requested except during a background run. Files may be given any names, example names are given only to match the truss example at the back of this manual.

Please input analysis type:

This option is for selecting modal analysis, CSI simulation or the transient response of a structure. Not all interactive inputs are required for each analysis type.

Do you wish to save an input file? (y or n)

This option creates a file which saves all the interactive input options. ACSIS can be background run with minor option changes by editing this file and then directing the screen input to this file. This file is very different for each analysis type. (To do this run with the example names after running acsis.exe once interactively, the command would be: acsis.exe <INP\_truss> &.

Name of save input file? (filename)

This question asks for the name under which to save the interactive input. example name: INP\_truss

Finite Element Model Input File Name (filename):

This file should contain all the finite element nodes, mesh, materials, properties, lumped inertias, fixations, and initial velocity and displacement conditions. It must be prepared in advance in the card format specified later. example name: FEM\_truss

Number of modes desired?

This only appears in the modal analysis to request the number of modes to be output. The actual analysis is carried out with double this number of modes for accuracy.

Controller Definition File Name:

This only appears for the CSI simulation. The file should contain the actuator and sensor locations, control gains, and observer gains. It also must be prepared in advance in card format. example name: CON\_truss

Please input type of control::

This option is for selecting the form of control law equations used in the CSI simulation. Full state feedback uses the current states of the plant (structure) to determine the control via constant gain matrices. Second-order observer uses only the L2 filter gain matrix of a Kalman filtering type of state estimation where the state variables are position and velocity. The Kalman filter option allows the use of a full set of filter gains, but the gain design must come from a alternate variable casting using position and generalized momenta.



Initial time, final time, control-on time, step size:

Data format is four columns for CSI, but only three columns for transient response since the control-on time part is deleted.

Forcing function ID, scale factor, damping coeff- a,b:

In order for any time dependent forcing function to be easily implemented despite variable time step sizes, the forcing functions must be entered into the subroutine forces.f. The format is to assign each new forcing function an ID number in a elseif statement. An example of forces.f is included at the back of the manual. Then forces.f must be recompiled into an object file and acsis.exe relinked. This is easily done by typing make which detects changes to the fortran files and does only the necessary compiling. A particular forcing function is chosen by entering its ID number in the first column, in addition the force can be scaled by a constant using the scaling factor in the second column. The Rayleigh damping coefficients a and b are the third and fourth columns.

Phase lag fix? (y or n):

Specifies whether to include an extra iteration of control and sensor state prediction at each integration step to improve accuracy. Not generally needed unless the user is investigating the source of instability in a simulation and needs to test the sensitivity of the response to the partitioned algorithm's extrapolation method.

Gain scale factors (4 total):

These scaling factors are, in order: the F1 control gain matrix (displacement), F2 control gain matrix (velocity), L1 and L2 state estimator filter gain matrix. This question appears only for CSI.

ACSIS has two types of output options. The first is the displacement or velocity motion of up to twenty separate degrees of freedom. Interactive questions are, 'Number of displacement results to output (max 10)' followed by as many 'Input node #, dof for displacement output #' as necessary. Then these repeat for velocity results. The name of the file where the output will be stored is requested by 'Output file name? (filename).' example name: OUT.truss. The second output option saves the displacement of all nodes at any time step where output is sent (see next entry). The format is suitable for animation of the entire structure. Question asks 'Animation Output? (y or n)' then for an animation filename if necessary.

Send output every how many steps?

This option affects both output options to reduce the size of the output and animation files. It causes output to only be sent after a integer number of time step iterations. To get output each time step, simply enter 1.

## Finite Element Input File

The finite element input file consists of title cards followed by columns of data. The title cards can be in any order, but they must be all capitalized with the appropriate number of columns of data for each card. The program reads rows of data until encountering a new card. Data which represents an integer value may be entered with a decimal point while real data may be entered without a decimal point as necessary. Any line beginning with a \* anywhere in the file is ignored and can be used to insert comments. Any blank line will result in a read error.

### NODES

Each node must be defined on a separate line. Columns can be separated by any number of spaces or a comma. Data format is four columns: node number, x-coordinate, y-coordinate, z-coordinate.

### TOPOLOGY

Each element must be defined on a separate line. Truss elements require two nodes then two columns of zeroes. (Truss elements would also require pin releases in ATTRIBUTES below.) Beam elements require two nodes then a third reference node representing a point in the xz plane of the beam and then a column of zeroes. Element type refers to finite element formulation. (Currently only type 1 = timeshenko beam element is now available.) Data format is six columns: element number, element type, node #1, node #2, node #3, node #4.

### ATTRIBUTES

Each element is characterized by an ID number from each of the MATERIAL and PROPERTIES cards below. Each element also has six pin release codes. The first code is for longitudinal stiffness, the second code is for torsional stiffness, the third and fifth codes are bending stiffness at each end in the y direction and must be the same value, and the fourth and sixth codes are for bending stiffness in the z direction and also must be the same. (0 is stiff, 1 is released.) Data format is nine columns: element number, material type, property type, and six pin release codes.

### MATERIAL

The material data is formatted in four columns: material type, Young's modulus, shear modulus and density of the material.

### PROPERTIES

The properties data is formatted in six columns: property type, cross sectional area of the element,  $I_y * I_x$ ,  $I_y$ ,  $I_x$ , shear shape factor SSF2, shear shape factor SSF3.

### FIXITY

Nodes with any fixations are defined here in the finite element file. The nodes must be entered with the fixity of all six of their DOF's, restrained or not. (1 is restrained, 0 is free) Data format is seven columns: node number, x, y, z,  $\phi_x$ ,  $\phi_y$ ,  $\phi_z$ .

## INERTIA

All lumped inertias must be entered with each separate DOF on an individual line. Therefore a single node could take up to six lines to define. Data format is three columns: node number, DOF number (1-6), and value of inertia.

## INITIAL CONDITIONS

All initial displacement and velocity conditions are entered into the finite element file. Data format is four columns: node number, DOF number (1-6), initial displacement, and initial velocity.

END

End of file.

## Controller Definition Cards

The controller definition file also consists of title cards followed by data entry. Each card must be followed by the appropriate number of columns of data and in some cases the appropriate number of rows. Integers can be entered in real format and vice versa if necessary. Any blank line will result in a read error.

NACT

Number of actuators in the entire control system.

EMAT

This entry creates the actuator position matrix or B matrix. There should be one row for each actuator, data format is four columns: node number, DOF number (1-6), actuator number, and sensitivity.

NSEN

Number of sensors in the entire controls system.

HDMA

This entry creates the matrix of displacement sensor locations. One row per displacement sensor, data format is four columns: node number, DOF number (1-6), sensor number, and sensitivity.

HVMA

This entry creates the matrix of velocity sensor locations. One row per velocity sensor, data format is four columns: node number, DOF number (1-6), sensor number, and sensitivity.

F1GA

This is a list of the F1 or displacement control gains. The data format is four columns: node number, DOF number (1-6), actuator number, and value of gain.

## F2GA

This is a list of the F2 or velocity control gains. The data format is four columns: node number, DOF number (1-6), actuator number, and value of gain.

## L1GA

This is a list of the state estimator L1 filter gains. The data format is four columns: node number, DOF number (1-6), actuator number, and value of gain.

## L2GA

This is a list of the state estimator L2 filter gains. The data format is four columns: node number, DOF number (1-6), actuator number, and value of gain.

END

End of file.

## Examples

This section includes all the files and procedures needed to run all three analysis on a simple elastic bar problem. The naming of the files is a simple and easy to remember system, however no particular format is necessary.

The finite element file was created simply by typing the node locations, connectivity (topology), etc. with a text editor.

File: FEM\_spring

---

```
*
MESH
*
NODES
 1  0.00 0.00 0.00
 2  1.00 0.00 0.00
 3  2.00 0.00 0.00
 4  3.00 0.00 0.00
TOPOLOGY
 1  1  1  2  0  0
 2  1  2  3  0  0
 3  1  3  4  0  0
ATTRIBUTES
 1  1  1  0  1  1  1  1  1
 2  1  1  0  1  1  1  1  1
 3  1  1  0  1  1  1  1  1
MATERIAL
 1 1000.  0.0  0.0
PROPERTIES
```

```
1 1.00 0.00 0.00 0.00 0.0 0.0
FIXITY
1 1 1 1 1 1
2 0 1 1 1 1
3 0 1 1 1 1
4 0 1 1 1 1
INERTIA
2 1 0.100
3 1 0.100
4 1 0.100
INITIAL
3 1 0.100 0.000
END
```

---

The modal analysis only requires the number of modes desired in addition to the finite element file. The file INP\_spring0 documents the interactive inputs used in the modal analysis. The results are saved in file EIG\_spring.

File: INP\_spring0

---

```
-1
n
* ACSIS input file,two lines above are
* analysis type and save input file. Do
* not change them by editing this file.
* Finite element input file?(filename)
FEM_spring
* Number of modes desired?
3
* Output file?(filename)
EIG_spring
```

---

File: EIG\_spring

---

#### SUBSPACE ITERATION ROUTINE

```
NB OF EIGENVALUES= 3
NB OF VECTOR= 3
NB OF DOF= 3
TOLERANCE= 1.000E-04

NB OF RIGID MODES= 0
```

```
ITERATION NO 1
```

```

1.000E+00  1.000E+00  1.000E+00
ITERATION NO  2
1.297E-14  3.194E-14  3.899E-14
EIGEN ANALYSIS RESULTS:

```

MODE	EIGENVALUE	RADIAL FREQUENCY	CYCLIC FREQUENCY
----	-----	-----	-----
1	1980.6	44.504	7.0831
2	15550.	124.70	19.846
3	32470.	180.19	28.679

EIGENVECTORS:

1	2.3305	1.8689	-1.0372	0.	0.
2	1.8689	-1.0372	2.3305	0.	0.
3	1.0372	-2.3305	-1.8689	0.	0.

MASS MATRIX DIAGONAL:

```

2 1 3  1.00000000000000D-01
3 1 2  1.00000000000000D-01
4 1 1  1.00000000000000D-01

```

To run the transient response of the structure, a forcing function or initial condition would be needed to excite the structure. An initial condition would be added to the finite element file. A forcing function must be added to forces.f with a new ID number, then this number given as interactive input. In this case, an initial displacement acts on the second degree of freedom, which is defined in the finite element model input file. The file INP\_spring1 documents the interactive inputs used in the transient analysis.

File: INP\_spring1

-3

n

```

* ACSIS input file,two lines above are
* analysis type and save input file. Do
* not change them by editing this file.
* Finite element input file?(filename)

```

FEM\_spring

```

* Initial, final, step size?
0.00000000  1.00000000  0.00100000
* Forcing function,scale f, damping a,b?
0  0.000000  0.00000000  0.00002000
* Output file name?(filename)

```

OUT\_spring

```

* Number of displacement outputs?
1
3 1
* Number of velocity outputs?

```

0

\* Send output every how many steps?

1

\* Send animation output?(y or n)

n

---

In addition to the finite element file, the interactive input, and an excitation, CSI simulation requires a controller definition file. A full state feedback controller for the truss structure is defined in CON\_spring.

File: CON\_spring

---

NACT

1

BMAT

3 1 1 1.0

F1GAIN

2 1 1 193.314150000000

3 1 1 1574.631500000000

4 1 1 -1669.046000000000

F2GAIN

2 1 1 20.77425600000000

3 1 1 27.79040300000000

4 1 1 10.78021200000000

NSEN

1

HVMAT

3 1 1 1.0

L1GAIN

2 1 1 8.79289090000000D-02

3 1 1 -4.88079010000000D-02

4 1 1 3.01027350000000D-03

L2GAIN

2 1 1 1.03809320000000D-01

3 1 1 6.14101300000000

4 1 1 -3.06087250000000

END

---

Then if acsis.exe is run interactively and the input is saved, the file INP\_spring2 can be produced. The file could be edited to change the input files, the output file, the forcing function ID, the length of simulation, control-on time, etc. Then to run it again, type acsis.exe<INP\_spring2> scr &. The scr is a scratch file which will store the screen output.

File: INP\_spring2

---

-2

n

- \* ACSIS input file,two lines above are
- \* analysis type and save input file. Do
- \* not change them by editing this file.
- \* Finite element input file?(filename)

FEM\_spring

- \* Controller file name?(filename)

CON\_spring

- \* Please input type of control:

3

- \* Initial,final,control-on,step size?

0.00000000 1.00000000 0.10000000 0.00100000

- \* Forcing function,scale f, damping a,b?

0 0.000000 0.00000000 0.00002000

- \* Phase lag fix?(y or n)

n

- \* Gain scale factors (4 total)?

1.00000000 1.00000000 1.00000000 1.00000000

- \* Output file name?(filename)

OUT\_spring

- \* Number of displacement outputs?

1

3 1

- \* Number of velocity outputs?

0

- \* Send output every how many steps?

1

- \* Send animation output?(y or n)

n

---



## APPENDIX B

Matlab Procedure and Scripts  
for Controller and Kalman Filter  
Gain Designs

## Introduction

In order to retain simplicity in the ACSIS code for parallel implementation purposes, no control system design algorithms were included. Instead, using modal data output from the eigenmode analysis module of ACSIS, a procedure was developed using the Pro-Matlab and its Control System Toolbox, which includes algorithm scripts for optimal control solutions via the solution of an algebraic Riccati equation. In order to accommodate large order dynamical systems, the design is accomplished in the uncoupled normal modes domain, using the available lowest eigenmodes from ACSIS.

The procedure begins by copying and editing the mode data output from ACSIS (see the listing for EIG\_spring in Appendix A) into readable variables for input to Matlab. The typical approach was to create one file as a Matlab script (i.e. a ".m" file), with the eigenvalues, eigenvectors, and mass/dof data from ACSIS at the beginning, followed by actuator and sensor influence matrices (related to the physical degrees of freedom), the objective function weighting matrices, and the function which calls other Matlab scripts to determine the solution. An example of the above input for the spring problem described in Appendix A is in file EIG\_spring.m, which is listed below. Compare this the the ACSIS mode data output shown in Appendix A to see how the editing was accomplished, and the additional control design data added

File: EIG\_spring.m

---

```
lam = [ 1980.6
        15550.
        32470. ];
v1=[1 2.3305 1.8689 -1.0372 0. 0.
     2 1.8689 -1.0372 2.3305 0. 0.
     3 1.0372 -2.3305 -1.8689 0. 0. ];
m=[ 2 1 3 1.00000000000000D-01
    3 1 2 1.00000000000000D-01
    4 1 1 1.00000000000000D-01];
t=[v1(:,2:4)];
qb=zeros(3,1);
qb(2,1)=1.0;
hd=zeros(1,3);
hv=zeros(1,3);
hv(1,2)=1.0;
qv=[1;.5;.1];
q=diag([qv;qv]);
r=.0001*eye(1);
[f1,f2]=mlqr(lam,t,m,qb,q,r,3);
qv=[1;1;1];
q=diag([qv;qv]);
r=100*eye(1);
[k1,k2]=mkf(lam,t,m,hd,hv,q,r,3);
save flout f1 /ascii
save f2out f2 /ascii
```

```
save k1out k1 /ascii
save k2out k2 /ascii
```

---

The scripts `mlqr.m` and `mkf.m` were written to accept as input the vector of eigenvalues, the eigenvector matrix (orthogonal vectors stored in columns), a matrix of node, component, d.o.f, mass data, and the actuator (or sensor) influence matrix and weighting matrices. The scripts output the gain results in four-column arrays, with one gain per row, and the corresponding node number, displacement component, and actuator (or sensor) identification. The top-level problem script (listed above) then saves the output in external files and the analysis is complete. The design scripts (listed below) also include checks on controllability and observability of the system based on the modal data and influence matrices defined, and produce plots of the closed loop poles resulting from the gain design. This aids the analyst in assessing the expected performance (and stability) of the exact system before moving the data back to ACSIS for simulation. The script `contrank.m` finds the rank of the controllability matrix through iterative rank calculations of submatrices so as to avoid the illconditioning experienced in the full matrix. This is both faster and more accurate for determining whether a particular actuator placement has full control of the included structural modes.

File: `mlqr.m`

---

```
function [F1out,F2out]=mlqr(lam,t,m,qb,Q,R,nmode)
%
% Controller gain design for second-order
% structural system via given eigenmodes.
% Gains are transformed to be coefficients
% of structural variables (disp,velocity);
% i.e. plant is second-order, of size ndof.
%
% Arguments:
%
% lam: vector of eigenvalues (nmode x 1)
% t: matrix of eigenvectors (ndof x nmode)
% m: mass diagonal and dof mapping info (ndof x 4)
% qb: actuator position influence matrix (nact x ndof)
% Q: optimal design state weighting matrix (2*nmode x 2*nmode)
% R: optimal design feedback weight. matrix (nact x nact)
%
% F1out: F1 gain matrix for 2nd-order plant
% F2out: F2 gain matrix for 2nd-order plant
%
% Written by K.F. Alvin
%
format short e
nmodmax=length(lam);
[ndof,nact]=size(qb);
%
% Variables:
% mass: mass matrix
```

```

% A: state transition matrix
% B: actuator influence matrix
% G: control gain matrix
%
massd(m(:,3))=m(:,4);
mass=diag(massd);
A=[zeros(nmode),eye(nmode);-1*diag(lam(1:nmode)),zeros(nmode)];
Amax=[zeros(nmodmax),eye(nmodmax);-1*diag(lam),zeros(nmodmax)];
B=[zeros(nmode,nact);t(:,1:nmode)*qb];
Bmax=[zeros(nmodmax,nact);t'*qb];
disp('Number of structural modes and actuators used:')
disp([nmode,nact])
disp('Rank of the controllability matrix:')
disp(conrank(A,B))
disp('Determining controller gains for given system...')
G=lqr(A,B,Q,R);
wmax=1.1*max(sqrt(lam));
axis([-wmax,wmax,-wmax,wmax]);
plot(eig(A-B*G),'*')
grid
title('Roots of controlled system')
hold
pause
%
% partition gain matrix
%
G1=G(:,1:nmode);
G2=G(:,nmode+1:nmode+nmode);
%
% Transform resultant gain matrices for use in
% partitioned csi algorithm using second-order
% structure(plant) equations.
%
disp('Mapping gains back to physical domain...')
f1=G1*t(:,1:nmode)*mass;
f2=G2*t(:,1:nmode)*mass;
%
% find modal damping ratios of controller for calculated gains:
%
Gmax=[f1*t,f2*t];
lambda=eig(Amax-Bmax*Gmax);
plot(lambda,'*')
pause
nfreq=sqrt(imag(lambda).^2 + real(lambda).^2);
mdamp=-real(lambda)./nfreq;
disp('Resultant modal damping ratios for controller:')
disp([' Damping ',' Damped Freq (rad/s) '])
disp([mdamp,nfreq.*sqrt(1-mdamp.^2),lambda])
bdamp=max(-2*mdamp./nfreq);
disp('Estimated minimum stiffness damping coefficient necessary')
disp(' to stabilize residual modes due to gain roundoff accumulation:')
disp(bdamp)
disp('Writing gains in node correspondence output form...')

```

```

Flout=zeros(nact*ndof,4);
F2out=zeros(nact*ndof,4);
for i=1:nact;
    kmin=(i-1)*ndof+1;
    kmax=i*ndof;
    Flout(kmin:kmax,:)= [m(:,1:2),i*ones(ndof,1),f1(i,m(:,3))'];
    F2out(kmin:kmax,:)= [m(:,1:2),i*ones(ndof,1),f2(i,m(:,3))'];
end;
disp('Finished mlqr')

```

---

File: mkf.m

---

```

function [L1out,L2out]=mkf(lam,t,m,hd,hv,Q,R,nmode)
%
% Kalman filter design for second-order
% structural system via given eigenmodes
% and transformed to independent
% displacement/gen. momentum variable
% casting for partitioned csi transient
% analysis. See Belvin/Park paper for
% filter variable definitions.
%
% Arguments:
%
% lam: vector of eigenvalues (nmode x 1)
% t: matrix of eigenvectors (ndof x nmode)
% m: mass diagonal and dof mapping info (ndof x 4)
% hd: sensor position influence matrix (nsen x ndof)
% hv: velocity position influence matrix (nsen x ndof)
% Q: optimal design state weighting matrix (2*nmode x 2*nmode)
% R: optimal design feedbk weight. matrix (nsen x nsen)
%
% L1out: L1 gain matrix for 2nd-order filter
% L2out: L2 gain matrix for 2nd-order filter
%
% Written by K.F. Alvin
%
format short e
nmodmax=length(lam);
[nsen,ndof]=size(hd);
%
% Variables:
% mass: mass matrix
% A: state transition matrix
% G: noise influence matrix
% C: output influence matrix
% K: filter gain matrix
%
massd(m(:,3))=m(:,4);
mass=diag(massd);
A=[zeros(nmode),eye(nmode);-1*diag(lam(1:nmode)),zeros(nmode)];
Amx=[zeros(nmodmax),eye(nmodmax);-1*diag(lam),zeros(nmodmax)];

```

```

G=eye(nmode+nmode);
C=[hd*t(:,1:nmode),hv*t(:,1:nmode)];
Cmax=[hd*t,hv*t];
disp('Number of structural modes and sensors used:')
disp([nmode,nsen])
disp('Rank of the observability matrix:')
disp(contrank(A',C'))
disp('Determining filter gains for given system...')
K=lqe(A,G,C,Q,R);
%
% partition gain matrix
%
K1=K(1:nmode,:);
K2=K(nmode+1:nmode+nmode,:);
Kmax=[K1;zeros(nmodmax-nmode,nsen);K2;zeros(nmodmax-nmode,nsen)];
%
% find modal damping ratios of filter for calculated gains:
%
lambda=eig(Amax-Kmax*Cmax);
plot(lambda,'+')
hold
pause
a1=-real(lambda);
b1=imag(lambda);
disp('Resultant modal damping ratios for filter:')
disp([' Damping ',' Freq (rad/s) '])
disp([a1./sqrt(a1.^2+b1.^2),b1])
%
% Transform resultant gain matrices for use in
% partitioned csi algorithm using second-order
% Kalman filter approach.
%
disp('Mapping gains back to physical domain...')
l1=t(:,1:nmode)*K1;
l2=mass*t(:,1:nmode)*K2;
disp('Writing gains in node correspondence output form...')
L1out=zeros(nsen*ndof,4);
L2out=zeros(nsen*ndof,4);
for i=1:nsen;
    kmin=(i-1)*ndof+1;
    kmax=i*ndof;
    L1out(kmin:kmax,:)= [m(:,1:2),i*ones(ndof,1),l1(m(:,3),i)];
    L2out(kmin:kmax,:)= [m(:,1:2),i*ones(ndof,1),l2(m(:,3),i)];
end;
disp('Finished mkf')

```

---

File: contrank.m

---

```

function maxrank=contrank(a,b)
maxrank=0;
[nstate,nact]=size(b);
i=0;

```

```

mat=b;
newrank=rank(mat);
while newrank > maxrank
  maxrank=newrank;
  i=i+1;
  mat=[mat,(a^-i)*b];
  newrank=rank(mat);
  if newrank==nstate
    maxrank=newrank;
  end
end
end

```

---

Unfortunately, the external files created from Matlab with the gain results are written completely in terms of real numbers, whereas the first three columns are actually to be read by ACSIS as integers (they are used as indices). A separate utility was written to convert the format of these files; the source code is listed below. On Unix systems, the user simply assigns the standard input to be the current data file created by Matlab, and gives another file name for the standard output. The code is basically just a filter to change the three columns of indices to integers. The output can then be pasted directly into the control definition file used by ACSIS.

File: convcont.f

---

```

program convcont

parameter (NMAX=100000)
real*8 f(NMAX),v(4)
integer node(NMAX),dof(NMAX),act(NMAX)

n=0

100 read (*,*,err=200,end=200) (v(i),i=1,4)
   n=n+1
   node(n)=int(v(1))
   dof(n) =int(v(2))
   act(n) =int(v(3))
   f(n)   =v(4)
   goto 100

200 do 300 i=1,n
   print *, node(i),dof(i),act(i),f(i)
300 continue

end

```

---

## APPENDIX C

### Stability Analysis of a CSI Partitioned Simulation Algorithm with State Estimator



The equation of the open-loop plant without passive damping in modal second-order form is

$$\ddot{q} + \omega^2 q = u \quad (1)$$

The controller uses a second-order observer to estimate the plant state, along with a full-state feedback control gain design.

$$\begin{aligned} u &= -(\eta\omega^2 p + \zeta\omega \dot{p}) \\ \ddot{p} + \omega^2 p &= u + \xi\gamma \\ \gamma &= z - \dot{p} \\ z &= \dot{q} \end{aligned} \quad (2)$$

where  $q$  and  $p$  are the plant and estimator states, respectively,  $u$  is the control force,  $\gamma$  is the state estimation error,  $z$  is the sensor output, and  $\eta, \zeta, \xi$  are gain coefficients for position and velocity feedback, and the estimator filter.

The partitioned analysis procedure uses a stabilized form of the control law and estimation error determination to reduce inaccuracies associated with the extrapolation of variables in the controller force prediction. A first-order filtering is achieved by taking the time derivative of (2a,c),

$$\begin{aligned} \dot{u} &= -\eta\omega^2 \dot{p} - \zeta\omega \ddot{p} \\ \dot{\gamma} &= \dot{z} - \ddot{p} \end{aligned} \quad (3)$$

and then embedding the equations of motion through substitution for  $\ddot{p}$ . This leads to the following two coupled, first-order differential equations for the prediction of the control force  $u$  and state error  $\gamma$ .

$$\begin{Bmatrix} \dot{u} \\ \dot{\gamma} \end{Bmatrix} + \begin{bmatrix} \zeta\omega & \zeta\xi\omega \\ 1 & \xi \end{bmatrix} \begin{Bmatrix} u \\ \gamma \end{Bmatrix} = \begin{Bmatrix} 0 \\ \dot{z} \end{Bmatrix} + \begin{Bmatrix} \zeta\omega^3 \\ \omega^2 \end{Bmatrix} p - \begin{Bmatrix} \eta\omega^2 \\ 0 \end{Bmatrix} \dot{p} \quad (4)$$

Time discretization of (4) using an implicit midpoint rule leads to the following coupled difference equation:

$$\begin{bmatrix} 1 + \delta\zeta\omega & \delta\zeta\xi\omega \\ \delta & 1 + \delta\xi \end{bmatrix} \begin{Bmatrix} u^{n+\frac{1}{2}} \\ \gamma^{n+\frac{1}{2}} \end{Bmatrix} = \begin{Bmatrix} 0 \\ z_p^{n+\frac{1}{2}} \end{Bmatrix} + \begin{Bmatrix} \delta\zeta\omega^3 - \eta\omega^2 \\ \delta\omega^2 \end{Bmatrix} p_p^{n+\frac{1}{2}} - \begin{Bmatrix} \zeta\omega \\ 1 \end{Bmatrix} \dot{p}^n \quad (5)$$

where  $\delta \equiv$  half-step size  $= \frac{h}{2}$ . Solving this equation requires knowledge of the plant (to obtain sensor output) and observer states. These values are extrapolated as:

$$\begin{aligned} p_p^{n+\frac{1}{2}} &= p^n + \delta\dot{p}^n \\ z_p^{n+\frac{1}{2}} &= \dot{q}_p^{n+\frac{1}{2}} = \dot{q}^n \end{aligned} \quad (6)$$

Using these equations to obtain  $u^{n+\frac{1}{2}}$  and  $\gamma^{n+\frac{1}{2}}$  allows the plant and observer equations to be solved independently. Midpoint time integration of (1) and (2b) leads to the following equations:

$$\begin{aligned}
 (1 + \delta^2 \omega^2) q^{n+\frac{1}{2}} &= \delta^2 u^{n+\frac{1}{2}} + q^n + \delta \dot{q}^n \\
 (1 + \delta^2 \omega^2) p^{n+\frac{1}{2}} &= \delta^2 u^{n+\frac{1}{2}} + p^n + \delta \dot{p}^n + \delta^2 \xi \gamma^{n+\frac{1}{2}} \\
 \dot{q}^{n+\frac{1}{2}} &= \frac{1}{\delta} (q^{n+\frac{1}{2}} - q^n) \\
 \dot{p}^{n+\frac{1}{2}} &= \frac{1}{\delta} (p^{n+\frac{1}{2}} - p^n) \\
 q^{n+1} &= 2q^{n+\frac{1}{2}} - q^n \\
 p^{n+1} &= 2p^{n+\frac{1}{2}} - p^n \\
 \dot{q}^{n+1} &= 2\dot{q}^{n+\frac{1}{2}} - \dot{q}^n \\
 \dot{p}^{n+1} &= 2\dot{p}^{n+\frac{1}{2}} - \dot{p}^n
 \end{aligned} \tag{7}$$

Computational stability of the modal form of the CSI partitioned equations of motion using the aforementioned time discretization can be assessed by seeking a nontrivial solution of

$$\begin{Bmatrix} q^{n+1} \\ p^{n+1} \\ \dot{q}^{n+1} \\ \dot{p}^{n+1} \end{Bmatrix} = \lambda \begin{Bmatrix} q^n \\ p^n \\ \dot{q}^n \\ \dot{p}^n \end{Bmatrix} \tag{8}$$

such that

$$|\lambda| \leq 1 \tag{9}$$

for stability. Substituting (8) into (5-7), we obtain

$$\mathbf{J}\mathbf{x} = 0 \tag{10}$$

where

$$\mathbf{x}_1 = [p_p^{n+\frac{1}{2}} \quad u^{n+\frac{1}{2}} \quad p^{n+\frac{1}{2}} \quad \dot{p}^{n+\frac{1}{2}} \quad p^n \quad \dot{p}^n]^T$$

$$\mathbf{x}_2 = [z_p^{n+\frac{1}{2}} \quad \gamma^{n+\frac{1}{2}} \quad q^{n+\frac{1}{2}} \quad \dot{q}^{n+\frac{1}{2}} \quad q^n \quad \dot{q}^n]^T$$

$$\mathbf{J} = \begin{bmatrix} \mathbf{J}_{11} & \mathbf{J}_{12} \\ \mathbf{J}_{21} & \mathbf{J}_{22} \end{bmatrix} \tag{11}$$

$$\mathbf{J}_{11} = \begin{bmatrix} 1 & 0 & 0 & 0 & -1 & -\delta \\ (\eta\omega^2 - \delta\zeta\omega^3) & (1 + \delta\zeta\omega) & 0 & 0 & 0 & \zeta\omega \\ 0 & -\delta^2 & (1 + \delta^2\omega^2) & 0 & -1 & -\delta \\ 0 & 0 & -\frac{1}{\delta} & 1 & \frac{1}{\delta} & 0 \\ 0 & 0 & -2 & \lambda + 1 & 0 & 0 \\ 0 & 0 & 0 & -2 & 0 & \lambda + 1 \end{bmatrix} \quad (12)$$

$$\mathbf{J}_{12} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & \delta\zeta\xi\omega & 0 & 0 & 0 & 0 \\ 0 & -\delta^2\xi & 0 & 0 & 0 & 0 \\ 0 & -\delta\xi & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \quad (13)$$

$$\mathbf{J}_{21} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ -\delta\omega^2 & \delta & 0 & 0 & 0 & 1 \\ 0 & -\delta^2 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \quad (14)$$

$$\mathbf{J}_{22} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & -1 \\ -1 & (1 + \delta\xi) & 0 & 0 & 0 & 0 \\ 0 & 0 & (1 + \delta^2\omega^2) & 0 & -1 & -\delta \\ 0 & 0 & -\frac{1}{\delta} & 1 & \frac{1}{\delta} & 0 \\ 0 & 0 & -2 & 0 & \lambda + 1 & 0 \\ 0 & 0 & 0 & -2 & 0 & \lambda + 1 \end{bmatrix} \quad (15)$$

A nontrivial solution to (10) is found from

$$\det \mathbf{J} = 0 \quad (16)$$

which leads to the characteristic equation

$$\begin{aligned} & ((1 - \delta\xi)(1 + \delta^3\zeta\omega^3 - \delta^2\eta\omega^2) + \delta\xi(1 + \delta^2\omega^2))z^4 \\ & + (\delta\zeta\omega(1 - \delta\xi) + \delta\xi(1 + \delta^3\zeta\omega^3 - \delta^2\eta\omega^2))z^3 \\ & ((1 - \delta\xi)(\delta^2\omega^2 + \delta^2\eta\omega^2) + \delta^2\omega^2(1 + \delta^3\zeta\omega^3 - \delta^2\eta\omega^2) \\ & + \delta\xi(\delta\zeta\omega + \delta^2\omega^2 + \delta^4\omega^4))z^2 \\ & (\delta^2\omega^2(\delta\zeta\omega) + \delta\xi(\delta^2\omega^2 + \delta^2\eta\omega^2))z \\ & + \delta^2\omega^2(\delta^2\omega^2 + \delta^2\eta\omega^2) = 0 \end{aligned} \quad (17)$$

where

$$\lambda = \frac{1+z}{1-z}, \quad |\lambda| \leq 1 \iff \operatorname{Re}(z) \leq 0 \quad (18)$$

Thus, a test of the polynomial equation for possible positive real roots by the Routh-Hurwitz criterion indicates that the partitioned approach as applied to the modal equations give a computationally stable solution for no velocity feedback  $\zeta = 0$  provided

$$h \leq \frac{2}{\sqrt{\eta}\omega} \quad (19)$$

$$h \leq \frac{2}{\xi} \quad (20)$$

**APPENDIX D**

**ACSIS Source Code**

File: Makefile

---

.SUFFIXES: .f .o

FFLAGS =

.f.o: fortran -c \$(FFLAGS) \*.f

OBJS = acsis.o acsisout.o addstf.o /  
beam3d.o forces.o input.o /  
pmvmul.o prepem.o profile.o /  
read.o solver.o nephlag.o /  
zerovect.o ln.o prepcon.o /  
control.o recorder.o measure.o /  
eigens.o singeig.o animout.o /  
stiffrc.o estifvm.o renum.o /  
kfilter.o

acsis.exe: \$(OBJS)  
fortran -o acsis.exe \$(FFLAGS) \$(OBJS)

---

File: shared.inc

---

C  
C -----  
C  
C shared.inc (ACSIS database)  
C  
C -----  
C  
C  
C Argument definitions:  
C adamp: Rayleigh damping coefficient alpha  
C b: actuator location matrix (packed storage)  
C brow: row number of corresponding real value in b  
C bcol: column number of corresponding real value in b  
C bval: number of nonzero values in b  
C bdamp: Rayleigh damping coefficient beta  
C confile: controller input file name  
C contype: id for type of control  
C coxyz: array of the x,y, and z components of each node  
C delta: one-half time step  
C delsq: one-half time step squared  
C ec: control prediction integration coefficient matrix  
C emat: array of element material types  
C eo: observer construct matrix S in vector form  
C eprop: array of element property types  
C es: structure construct matrix S ( $M+\delta*D+\delta^2*K$ )  
C etype: array of element types  
C elnum: array of element numbers for domain decomposition  
C f: vector of applied forces  
C fi: control gain matrix

C f2: control gain matrix  
C femfile: finite element input file name  
C forceid: identification number of forcing function  
C gamma: state correction force  
C gc: RHS vector for control prediction module  
C gk: RHS vector for Kalman filter momentum eqn  
C go: RHS vector for observer module  
C gs: RHS vector for structure module  
C h: time step size  
C hd: displacement sensor location matrix (packed storage)  
C hdrow: row number of corresponding real value in hd  
C hdcol: column number of corresponding real value in hd  
C hdval: number of nonzero values in hd  
C hv: velocity sensor location matrix (packed storage)  
C hvrow: row number of corresponding real value in hv  
C hvcol: column number of corresponding real value in hv  
C hvval: number of nonzero values in hv  
C id: DOF mapping array: id(comp,node #)=Global DOF #  
C ix: array of element connectivity and orientation  
C inertia: array of concentrated inertias or lumped masses  
C jdiag: array of diagonal element addresses  
C l1: State estimator filter gain matrix  
C l2: State estimator filter gain matrix  
C mask:  
C mass: mass matrix M in reduced vector form  
C mat: array of different materials  
C mlen: length of global matrices in profile vector storage  
C nact: actual number of actuators  
C ncsi: actual number of actuators and sensors  
C ndisout: number of displacement results to output  
C ndof: actual number of degrees of freedom  
C ndomain: actual number of element domains (for dom. decomp.)  
C nel: actual number of elements  
C neld: array of number of elements in each domain  
C nnp: actual number of nodes  
C nsen: actual number of sensors  
C nvelout: number of velocity results to output  
C nolag: logical flag to signal corrector loop in measurement  
C outfile: output file name  
C outlabel: array of output data requested  
C pin: array of element pin release codes  
C pivot: Column pivoting info from FACTA  
C prop: array of different properties  
C q: generalized displacement vector  
C q0: initial displacement condition  
C qalpha: gain scale factor for f1  
C qalphao: gain scale factor for l1  
C qbeta: gain scale factor for f2  
C qbetao: gain scale factor for l2  
C qdot: velocity vector  
C qdot0: initial velocity condition  
C qe: state estimator displacement vector  
C qedot: state estimator velocity vector  
C r: Solution vector of control module {u,gamma}  
C scalef: Scaling factor for forcing function  
C stiff: stiffness matrix K in reduced vector form  
C t0: initial time  
C tc: control-on time  
C tf: final time

C  
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C

u: vector of control forces

Parameter definitions

MAXACT: max. # of actuators  
MAXCSI: max. combined # of actuators and sensors  
MAXDAT: max. # of materials and properties  
MAXDOF: max. # of degrees of freedom  
MAXDOM: max. # of decomposition domains  
MAXELE: max. # of elements  
MAXMLN: max. length of global vectors in reduced form  
MAXNODE: max. # of nodes

parameter(MAXDOF=3000, MAXACT=50, MAXCSI=100)  
parameter(MAXNODE=1000, MAXELE=3000, MAXDAT=100)  
parameter(MAXMLN=200000, MAXDOM=50, MAXNZV=200)

real\*8 t0,tf,tc,h,delta,delsq,qalpha,qbeta,qalphao,qbetao  
real\*8 q(MAXDOF),qdot(MAXDOF),qe(MAXDOF),qedot(MAXDOF)  
real\*8 u(MAXACT),gamma(MAXACT),f(MAXDOF),r(MAXCSI)  
real\*8 es(MAXMLN),eo(MAXMLN),ec(MAXCSI,MAXCSI)  
real\*8 gs(MAXDOF),go(MAXDOF),gc(MAXCSI),scalef,pe(MAXDOF)  
real\*8 mass(MAXMLN),stif(MAXMLN),adamp,bdamp,gk(MAXDOF)  
real\*8 coxyz(3,MAXNODE),mat(6,MAXDAT),prop(10,MAXDAT)  
real\*8 q0(6,MAXNODE),qdot0(6,MAXNODE),inertia(6,MAXNODE)  
real\*8 b(MAXNZV),hd(MAXNZV),hv(MAXNZV),estifm(78,500)  
real\*8 f1(MAXACT,MAXDOF),f2(MAXACT,MAXDOF)  
real\*8 l1(MAXDOF,MAXACT),l2(MAXDOF,MAXACT)  
integer etype(MAXELE),ix(4,MAXELE),emat(MAXELE),forceid  
integer eprop(MAXELE),pin(6,MAXELE),id(6,MAXNODE)  
integer mask(MAXNODE),contype,brow(MAXNZV),bcol(MAXNZV)  
integer hdrow(MAXNZV),hdcol(MAXNZV),hvrow(MAXNZV)  
integer hvcol(MAXNZV),bval,hdval,hvval  
integer ndof,nact,nsen,ncsi,mLen,jdiag(MAXDOF),nnp,nel,neig  
integer neld(MAXDOM),elnum(MAXELE,MAXDOM),eldom(MAXELE),ndomain  
integer outlabel(40),ndisout,nvelout,pivot(MAXCSI)  
integer iadjcy(MAXMLN),icount(MAXNODE+1),perm(MAXNODE)  
integer xls(MAXNODE)  
logical animate,nolag  
character\*32 femfile,confile,outfile,animfile

COMMON /FILES/ femfile,confile,outfile,outlabel,ndisout,  
nvelout,animfile,animate,nolag  
COMMON /TIMERS/ t0,tf,tc,h,delta,delsq  
COMMON /STATES/ q,qdot,qe,qedot,u,gamma,f,r,pe  
COMMON /FENDAT/ mass,stif,adamp,bdamp,coxyz,mat,prop,q0,qdot0,  
inertia,scalef  
COMMON /INTEGR/ es,eo,ec,gs,go,gc,gk,pivot  
COMMON /DIMENS/ ndof,nact,nsen,ncsi,mLen,jdiag,nnp,nel,neig  
COMMON /CONDAT/ b,hd,hv,f1,f2,l1,l2,  
qalpha,qbeta,qalphao,qbetao  
COMMON /ELEDOM/ estifm,ndomain,neld,elnum,eldom  
COMMON /INTGER/ forceid,etype,ix,emat,eprop,pin,id,mask,  
contype,brow,bcol,hdrow,hdcol,hvrow,hvcol,  
bval,hdval,hvval  
COMMON /RESEQN/ iadjcy,icount,perm,xls

File: acsis.f



C=Program ACSIS  
C=Purpose Accelerated CSI Simulation  
C=Author K. Alvin  
C=Date May 1990  
C=Block Fortran

```
C-----C  
C  
C   program ACSIS  
C  
C  
C   Purpose:   2nd Order Accelerated CSI Simulation  
C  
C  
C-----C
```

```
   program ACSIS  
  
C   GET SHARED DATA FILE  
  
   include 'shared.inc'  
  
C   LOCAL VARIABLES  
  
   real*8 t,z(MAXIACT)  
  
   integer n,m,runtime,outskip  
  
C   LOGIC  
  
   call INPUT(runtime,outskip)  
  
   if (runtime) 100,200,300  
  
C   EIGENMODE ANALYSIS  
  
100  continue  
   call PREPFEM  
   call EIGENS  
   goto 999  
  
C   CSI SIMULATION  
  
200  call PREPFEM  
   call PREPCON  
  
   n = 0  
   m = 0  
   print *, 'Finished Preprocessing . . . starting simulation'  
   print *, 'Time =',t0  
   CALL ACSISOUT(t0)  
  
   do 250 t=t0,tf,h  
  
       call FORCES(t+h/2)  
  
C   Predict CSI coupling variables u and gamma
```

```

    if (t .ge. tc) then
      call MEASURE(z)
      call CONTROL(z)
      if (nolag) then
        call NOPHLAG(z)
        call CONTROL(z)
      endif
    endif

C   Structure and observer set up for parallel execution

CVD$  CNCALL
      do 275 i=1,2

C   Integrate Observer Equations

      if ((i .eq. 1).and.(t .ge. tc)) then
        if (contype .eq. 0) then
          call SECORDER(mass,stif,adamp,bdamp,f,go,eo,qe,qdot,
            delta,delsq,jdiag,ndof,MAXDOF)
        elseif (contype .eq. 1) then
          call KFILTER
        endif

C   Integrate Structure Equations

        elseif (i .eq. 2) then
          call SECORDER(mass,stif,adamp,bdamp,f,gs,es,q,qdot,
            delta,delsq,jdiag,ndof,MAXDOF)
        endif

275    continue

C   PRINT TIME EACH 100 iterations

      n = n + 1
      m = m + 1
      if (n .ge. 100) then
        print *, 'Time = ',t+h
        n = 0
      endif
      if (m .ge. outskip) then
        call ACSISOUT(t+h)
        write(24,'(40f12.8)') t,(z(i),i=1,nsen)
        m = 0
      endif

250    continue
      goto 999

C   TRANSIENT RESPONSE

300    call PREPFEM

      n = 0
      m = 0
      print *, 'Finished Preprocessing . . . starting simulation'
      print *, 'Time = ',t0

```

```

call ACSISOUT(t0)

do 350 t=t0,tf,h

    call FORCES(t+h/2)

    call ZEROVECT(gs,ndof)

    call RECORDER(mass,stif,adamp,bdamp,f,gs,es,q,qdot,
                  delta,delsq,jdiag,ndof,MAXDOF)

C   PRINT TIME EACH 100 iterations

    n = n + 1
    m = m + 1
    if (n .ge. 100) then
        print *, 'Time = ',t+h
        n = 0
    endif
    if (m .ge. outskip) then
        call ACSISOUT(t+h)
        m = 0
    endif

350  continue

999  stop
     end

```

---

File: acsisout.f

---

```

C=Module ACSISOUT
C=Purpose Write desired output from ACSIS for plotting, etc.
C=Author K. Alvin
C=Date May 1990
C=Block Fortran
C -----
C
C   Subroutine ACSISOUT
C
C   Purpose:
C       This subroutine outputs formatted displacement and velocity
C       results at a given time for plotting time histories. The
C       desired output variables are defined in outlabel().
C -----
C
C   Arguments
C       t - time
C
subroutine ACSISOUT(t)

include 'shared.inc'
real*8 t

```

```

C   LOCAL VARIABLES

      integer i

C   LOGIC

      write(13,'(40f12.8)') t,(q(id(outlabel(i+10),outlabel(i))),
.   i=1,ndisout),(qdot(id(outlabel(i+30),outlabel(i+20))),
.   i=1,nvelout)

      write(23,'(40f12.8)') t,(qa(id(outlabel(i+10),outlabel(i))),
.   i=1,ndisout),(qdot(id(outlabel(i+30),outlabel(i+20))),
.   i=1,nvelout)

      write(25,'(40f12.8)') t,(u(i),i=1,nact),(gamma(i),i=1,nsen)

      if (animate) call ANIMOUT(q,id,nnp,t,15)

      return
      end

```

File: addstf.f

```

C=Module ADDSTF
C=Purpose Assemble Global stiffness matrix
C=Author who knows
C=Update January 1989, by E. Pramono
C=Block Fortran
      subroutine ADDSTF(sk,lm,bk,jdiag,nseq)
C-----+C
C   PURPOSE:
C   THIS SUBROUTINE ASSEMBLES THE ELEMENT STIFFNESS MATRICES
C   INTO THE COMPACTED GLOBAL STIFFNESS VECTOR.
C
C   ARGUMENTS:
C   sk - ELEMENT STIFFNESS MATRIX
C   lm - LOCATION VECTOR FOR ELEMENT STIFFNESS MATRIX
C   bk - COMPACTED GLOBAL STIFFNESS VECTOR
C   jdiag - VECTOR OF DIAGONAL ELEMENT ADDRESSES
C   nseq - NUMBER OF DEGREES OF FREEDOM PER ELEMENT
C-----C

C   ARGUMENTS

      real*8 sk(nseq,nseq), bk(1)
      integer lm(18), jdiag(1)
C   integer lm(18), jdiag(1), nseq

C   LOCAL ARGUMENTS

      integer i, j, k, l, m

C   ASSEMBLE GLOBAL STIFFNESS AND LOAD ARRAYS

      do 20 j = 1, nseq
         k = lm(j)
         if (k .eq. 0) goto 20

```

```

      l = jdiag(k) - k
C     l = jdiag(k+1) - k
      do 10 i = 1, nseq
          m = lm(i)
          if(m .gt. k .OR. m .eq. 0) goto 10
          m = l + m
          bk(m) = bk(m) + sk(i,j)
10     continue
20     continue
C
      return
      end
C=End Fortran

```

File: beam3d.f

```

C=Module BEAM3D
C=Purpose Construct 3-d Timoshenko beam element stiffness and lumped mass
C=Author K. Alvin
C=Date May 1990
C=Block Fortran

```

```

      subroutine BEAM3D(n,ni,nj,nk,xyz,emod,gmod,rho,area,ssf2,ssf3,
&                      jtor,i2,i3,ipin,sk,sm)

```

C ARGUMENTS:

```

C
C     n      Element ID Number
C     ni     Node ID Number at End i
C     nj     Node ID Number at End j
C     xyz    Node Location Array
C     emod   Material Elastic Modulus (Young's Modulus)
C     gmod   Material Modulus of Rigidity (Shear Modulus)
C     rho    Material Mass Density
C     area   Element Cross-sectional area
C     ssf2   Shear shape factor in element x2 direction
C     ssf3   Shear shape factor in element x3 direction
C     jtor   Torsional constant J
C     i2     Area moment of inertia about element x2 axis
C     i3     Area moment of inertia about element x3 axis
C     ipin   Pin release codes: 0=Fixed, 1=Freed
C           (1) Axial
C           (2) Torsional
C           (3) End A rotation about x2 axis
C           (4) End A rotation about x3 axis
C           (5) End B rotation about x2 axis
C           (6) End B rotation about x3 axis
C     sk     Element Stiffness Matrix
C     sm     Element Mass Matrix

```

```

integer n,ni,nj,nk,ipin(1)
real*8 xyz(3,1),emod,gmod,rho,area,ssf2,ssf3,jtor,i2,i3
real*8 sk(12,1),sm(12,1)

```

C LOCAL VARIABLES:

```

integer i,j
real*8 dc(3,3),length,rlength,kc(10),mc(3)

```

C LOGIC

C Find Element Length

```
length = 0.0d0
do 10 i = 1,3
  dc(1,i) = xyz(i,nj) - xyz(i,ni)
  length = length + dc(1,i)**2
10 continue
length = sqrt(length)
if (length .eq. 0.0d0) then
  print *, 'BAR2D: Zero element length; n= ',n
  return
endif
```

C Find direction cosines for x1,x2,x3 element axes

```
do 15 i=1,3
  dc(1,i) = dc(1,i)/length
  if (nk .eq. 0) then
    dc(2,i) = 0.0
  else
    dc(2,i) = xyz(i,nk) - xyz(i,ni)
  endif
15 continue
if (nk .eq. 0) dc(2,3) = 1.0
16 dc(3,1) = dc(1,2)*dc(2,3) - dc(2,2)*dc(1,3)
dc(3,2) = dc(2,1)*dc(1,3) - dc(1,1)*dc(2,3)
dc(3,3) = dc(1,1)*dc(2,2) - dc(2,1)*dc(1,2)
rlength = sqrt(dc(3,1)**2 + dc(3,2)**2 + dc(3,3)**2)
if (rlength .ne. 0.) goto 17
dc(2,2) = 1.0
dc(2,3) = 0.0
goto 16

17 do 18 i=1,3
  dc(3,i) = dc(3,i)/rlength
18 continue
dc(2,1) = dc(3,2)*dc(1,3) - dc(1,2)*dc(3,3)
dc(2,2) = dc(1,1)*dc(3,3) - dc(3,1)*dc(1,3)
dc(2,3) = dc(3,1)*dc(1,2) - dc(1,1)*dc(3,2)
```

C Compute various stiffness constants, accounting for pin codes

```
if (ipin(1) .eq. 0) then
  kc(1) = area*emod/length
else
  kc(1) = 0.0d0
endif

if (ipin(4) .eq. 0) then
  if (ipin(6) .eq. 0) then
    kc(2) = area*gmod*ssf2/length
    kc(6) = i3*emod/length
    kc(7) = kc(2)*length/2.0d0
    kc(9) = kc(7)*length/2.0d0
  else
    print *, 'BEAM3D: Pin code error, x3 direction, el #',n
```

```

endif
else
if (ipin(6) .eq. 0) then
print *, 'BEAN3D: Pin code error, x3 direction, el #', n
else
kc(2) = 0.0d0
kc(6) = 0.0d0
kc(7) = 0.0d0
kc(9) = 0.0d0
endif
endif

if (ipin(3) .eq. 0) then
if (ipin(5) .eq. 0) then
kc(3) = area*gmod*ss13/length
kc(5) = i2*emod/length
kc(8) = kc(3)*length/2.0d0
kc(10) = kc(8)*length/2.0d0
else
print *, 'BEAN3D: Pin code error, x2 direction, el #', n
endif
endif

else
if (ipin(5) .eq. 0) then
print *, 'BEAN3D: Pin code error, x2 direction, el #', n
else
kc(3) = 0.0d0
kc(5) = 0.0d0
kc(8) = 0.0d0
kc(10) = 0.0d0
endif
endif

endif

if (ipin(2) .eq. 0) then
kc(4) = jtor*gmod/length
else
kc(4) = 0.0d0
endif

mc(1) = area*rho*length/2.0d0
mc(2) = i2*rho*length/2.0d0
mc(3) = i3*rho*length/2.0d0

sk(1,1) = kc(1)*dc(1,1)*dc(1,1) + kc(2)*dc(2,1)*dc(2,1) +
          kc(3)*dc(3,1)*dc(3,1)
sk(1,2) = kc(1)*dc(1,1)*dc(1,2) + kc(2)*dc(2,1)*dc(2,2) +
          kc(3)*dc(3,1)*dc(3,2)
sk(1,3) = kc(1)*dc(1,1)*dc(1,3) + kc(2)*dc(2,1)*dc(2,3) +
          kc(3)*dc(3,1)*dc(3,3)
sk(1,4) = kc(7)*dc(2,1)*dc(3,1) - kc(8)*dc(3,1)*dc(2,1)
sk(1,5) = kc(7)*dc(2,1)*dc(3,2) - kc(8)*dc(3,1)*dc(2,2)
sk(1,6) = kc(7)*dc(2,1)*dc(3,3) - kc(8)*dc(3,1)*dc(2,3)
sk(1,7) = -sk(1,1)
sk(1,8) = -sk(1,2)
sk(1,9) = -sk(1,3)
sk(1,10) = sk(1,4)
sk(1,11) = sk(1,5)
sk(1,12) = sk(1,6)
sk(2,2) = kc(1)*dc(1,2)*dc(1,2) + kc(2)*dc(2,2)*dc(2,2) +
          kc(3)*dc(3,2)*dc(3,2)

```

$sk(2,3) = kc(1)*dc(1,2)*dc(1,3) + kc(2)*dc(2,2)*dc(2,3) + kc(3)*dc(3,2)*dc(3,3)$   
 $sk(2,4) = kc(7)*dc(2,2)*dc(3,1) - kc(8)*dc(3,2)*dc(2,1)$   
 $sk(2,5) = kc(7)*dc(2,2)*dc(3,2) - kc(8)*dc(3,2)*dc(2,2)$   
 $sk(2,6) = kc(7)*dc(2,2)*dc(3,3) - kc(8)*dc(3,2)*dc(2,3)$   
 $sk(2,7) = -sk(1,2)$   
 $sk(2,8) = -sk(2,2)$   
 $sk(2,9) = -sk(2,3)$   
 $sk(2,10) = sk(2,4)$   
 $sk(2,11) = sk(2,5)$   
 $sk(2,12) = sk(2,6)$   
 $sk(3,3) = kc(1)*dc(1,3)*dc(1,3) + kc(2)*dc(2,3)*dc(2,3) + kc(3)*dc(3,3)*dc(3,3)$   
 $sk(3,4) = kc(7)*dc(2,3)*dc(3,1) - kc(8)*dc(3,3)*dc(2,1)$   
 $sk(3,5) = kc(7)*dc(2,3)*dc(3,2) - kc(8)*dc(3,3)*dc(2,2)$   
 $sk(3,6) = kc(7)*dc(2,3)*dc(3,3) - kc(8)*dc(3,3)*dc(2,3)$   
 $sk(3,7) = -sk(1,3)$   
 $sk(3,8) = -sk(2,3)$   
 $sk(3,9) = -sk(3,3)$   
 $sk(3,10) = sk(3,4)$   
 $sk(3,11) = sk(3,5)$   
 $sk(3,12) = sk(3,6)$   
 $sk(4,4) = kc(4)*dc(1,1)*dc(1,1) + (kc(10)+kc(5))*dc(2,1)*dc(2,1) + (kc(9)+kc(6))*dc(3,1)*dc(3,1)$   
 $sk(4,5) = kc(4)*dc(1,1)*dc(1,2) + (kc(10)+kc(5))*dc(2,1)*dc(2,2) + (kc(9)+kc(6))*dc(3,1)*dc(3,2)$   
 $sk(4,6) = kc(4)*dc(1,1)*dc(1,3) + (kc(10)+kc(5))*dc(2,1)*dc(2,3) + (kc(9)+kc(6))*dc(3,1)*dc(3,3)$   
 $sk(4,7) = -sk(1,4)$   
 $sk(4,8) = -sk(2,4)$   
 $sk(4,9) = -sk(3,4)$   
 $sk(4,10) = -kc(4)*dc(1,1)*dc(1,1) + (kc(10)-kc(5))*dc(2,1)*dc(2,1) + (kc(9)-kc(6))*dc(3,1)*dc(3,1)$   
 $sk(4,11) = -kc(4)*dc(1,1)*dc(1,2) + (kc(10)-kc(5))*dc(2,1)*dc(2,2) + (kc(9)-kc(6))*dc(3,1)*dc(3,2)$   
 $sk(4,12) = -kc(4)*dc(1,1)*dc(1,3) + (kc(10)-kc(5))*dc(2,1)*dc(2,3) + (kc(9)-kc(6))*dc(3,1)*dc(3,3)$   
 $sk(5,5) = kc(4)*dc(1,2)*dc(1,2) + (kc(10)+kc(5))*dc(2,2)*dc(2,2) + (kc(9)+kc(6))*dc(3,2)*dc(3,2)$   
 $sk(5,6) = kc(4)*dc(1,2)*dc(1,3) + (kc(10)+kc(5))*dc(2,2)*dc(2,3) + (kc(9)+kc(6))*dc(3,2)*dc(3,3)$   
 $sk(5,7) = -sk(1,5)$   
 $sk(5,8) = -sk(2,5)$   
 $sk(5,9) = -sk(3,5)$   
 $sk(5,10) = sk(4,11)$   
 $sk(5,11) = -kc(4)*dc(1,2)*dc(1,2) + (kc(10)-kc(5))*dc(2,2)*dc(2,2) + (kc(9)-kc(6))*dc(3,2)*dc(3,2)$   
 $sk(5,12) = -kc(4)*dc(1,2)*dc(1,3) + (kc(10)-kc(5))*dc(2,2)*dc(2,3) + (kc(9)-kc(6))*dc(2,3)*dc(3,3)$   
 $sk(6,6) = kc(4)*dc(1,3)*dc(1,3) + (kc(10)+kc(5))*dc(2,3)*dc(2,3) + (kc(9)+kc(6))*dc(3,3)*dc(3,3)$   
 $sk(6,7) = -sk(1,6)$   
 $sk(6,8) = -sk(2,6)$   
 $sk(6,9) = -sk(3,6)$   
 $sk(6,10) = sk(4,12)$   
 $sk(6,11) = sk(5,12)$   
 $sk(6,12) = -kc(4)*dc(1,3)*dc(1,3) + (kc(10)-kc(5))*dc(2,3)*dc(2,3) + (kc(9)-kc(6))*dc(3,3)*dc(3,3)$   
 $sk(7,7) = sk(1,1)$



```

sk(7,8) = sk(1,2)
sk(7,9) = sk(1,3)
sk(7,10) = -sk(1,4)
sk(7,11) = -sk(1,5)
sk(7,12) = -sk(1,6)
sk(8,8) = sk(2,2)
sk(8,9) = sk(2,3)
sk(8,10) = -sk(2,4)
sk(8,11) = -sk(2,5)
sk(8,12) = -sk(2,6)
sk(9,9) = sk(3,3)
sk(9,10) = -sk(3,4)
sk(9,11) = -sk(3,5)
sk(9,12) = -sk(3,6)
sk(10,10) = sk(4,4)
sk(10,11) = sk(4,5)
sk(10,12) = sk(4,6)
sk(11,11) = sk(5,5)
sk(11,12) = sk(5,6)
sk(12,12) = sk(6,6)

```

```

do 50 i = 1,12
  do 55 j = 1,12
    sm(i,j) = 0.d0
  55 continue
50 continue

```

55  
50

C Row-sum rotated mass matrix to re-diagonalize

```

sm(1,1) = mc(1)
sm(2,2) = mc(1)
sm(3,3) = mc(1)
sm(4,4) = mc(2)*(dc(1,1)*dc(1,1)+dc(2,1)*dc(2,1)) +
.         mc(3)*(dc(1,1)*dc(1,1)+dc(3,1)*dc(3,1)) +
.         mc(2)*(dc(1,1)*dc(1,2)+dc(2,1)*dc(2,2)) +
.         mc(3)*(dc(1,1)*dc(1,2)+dc(3,1)*dc(3,2)) +
.         mc(2)*(dc(1,1)*dc(1,3)+dc(2,1)*dc(2,3)) +
.         mc(3)*(dc(1,1)*dc(1,3)+dc(3,1)*dc(3,3))
C sm(4,5) = mc(2)*(dc(1,1)*dc(1,2)+dc(2,1)*dc(2,2)) +
C .         mc(3)*(dc(1,1)*dc(1,2)+dc(3,1)*dc(3,2))
C sm(4,6) = mc(2)*(dc(1,1)*dc(1,3)+dc(2,1)*dc(2,3)) +
C .         mc(3)*(dc(1,1)*dc(1,3)+dc(3,1)*dc(3,3))
C sm(5,4) = mc(2)*(dc(1,1)*dc(1,2)+dc(2,1)*dc(2,2)) +
C .         mc(3)*(dc(1,1)*dc(1,2)+dc(3,1)*dc(3,2))
sm(5,5) = mc(2)*(dc(1,2)*dc(1,2)+dc(2,2)*dc(2,2)) +
.         mc(3)*(dc(1,2)*dc(1,2)+dc(3,2)*dc(3,2)) +
.         mc(2)*(dc(1,1)*dc(1,2)+dc(2,1)*dc(2,2)) +
.         mc(3)*(dc(1,1)*dc(1,2)+dc(3,1)*dc(3,2)) +
.         mc(2)*(dc(1,2)*dc(1,3)+dc(2,2)*dc(2,3)) +
.         mc(3)*(dc(1,2)*dc(1,3)+dc(3,2)*dc(3,3))
C sm(5,6) = mc(2)*(dc(1,2)*dc(1,3)+dc(2,2)*dc(2,3)) +
C .         mc(3)*(dc(1,2)*dc(1,3)+dc(3,2)*dc(3,3))
C sm(6,4) = mc(2)*(dc(1,1)*dc(1,3)+dc(2,1)*dc(2,3)) +
C .         mc(3)*(dc(1,1)*dc(1,3)+dc(3,1)*dc(3,3))
C sm(6,5) = mc(2)*(dc(1,2)*dc(1,3)+dc(2,2)*dc(2,3)) +
C .         mc(3)*(dc(1,2)*dc(1,3)+dc(3,2)*dc(3,3))
sm(6,6) = mc(2)*(dc(1,3)*dc(1,3)+dc(2,3)*dc(2,3)) +
.         mc(3)*(dc(1,3)*dc(1,3)+dc(3,3)*dc(3,3)) +

```

```

      mc(2)*(dc(1,1)*dc(1,3)+dc(2,1)*dc(2,3)) +
      mc(3)*(dc(1,1)*dc(1,3)+dc(3,1)*dc(3,3)) +
      mc(2)*(dc(1,2)*dc(1,3)+dc(2,2)*dc(2,3)) +
      mc(3)*(dc(1,2)*dc(1,3)+dc(3,2)*dc(3,3))
sm(7,7) = mc(1)
sm(8,8) = mc(1)
sm(9,9) = mc(1)
sm(10,10) = sm(4,4)
C sm(10,11) = sm(4,5)
C sm(10,12) = sm(4,6)
C sm(11,10) = sm(5,4)
sm(11,11) = sm(5,5)
C sm(11,12) = sm(5,6)
C sm(12,10) = sm(6,4)
C sm(12,11) = sm(6,5)
sm(12,12) = sm(6,6)

do 100 i=1,12
  do 200 j=1,i-1
    sk(i,j) = sk(j,i)
200  continue
100  continue

return
end
C=End Fortran

```

---

File: forces.f

---

```

C=Module FORCES
C=Purpose Calculate applied force vector at given time
C=Author K. Alvin
C=Date May 1990
C=Block Fortran
C -----
C
C Subroutine FORCES
C
C Purpose:
C Returns force from stored function at any given time.
C The forcing functions are hardwired by the user. The
C function is selectable at program execution using the
C forcing function ID, which by convention is the statement
C label used in branching.
C -----
C
C
C subroutine FORCES(time)
C
C include 'shared.inc'
C real*8 time,pi
C
C LOGIC
C
C pi = 3.1415926

```

```

call ZEROVECT(f,ndof)

101 if (forceid .eq. 101) then
    if (time .le. .02) then
        f(id(2,15)) = 100.*(1.-cos(2.*pi*time/.02))
    endif

102 elseif (forceid .eq. 102) then
    if (time .lt. .1) then
        f(id(2,95)) = 10.
    elseif (time .eq. .1) then
        f(id(2,95)) = 0.
    elseif ((time .gt. .1).and.(time .lt. .2)) then
        f(id(2,95)) = -10.
    else
        f(id(2,95)) = 0.
    endif

103 elseif (forceid .eq. 103) then
    if (time .le. .01) then
        f(id(1,15)) = 100
    endif

104 elseif (forceid .eq.104) then
    if ((time .gt. 0) .and. (time .lt. .17)) then
        f(id(3,125)) = 10
    elseif ((time .gt. .17) .and. (time .lt. 1.0)) then
        f(id(3,125)) = -10
    endif

105 elseif (forceid .eq. 105) then
    if (time .le. .01) then
        f(id(2,9)) = 100.*(1.-cos(2.*pi*time/.01))
    elseif (time .le. .02) then
        f(id(2,9)) = 100.*(cos(2.*pi*time/.01)-1.)
    endif

endif

do 10 i = 1, ndof
    f(i) = scalef * f(i)
10 continue

return
end

```

---

File: input.f

---

C=Module INPUT  
C=Purpose Input data parameters for ACSIS  
C=Author K. Alvin

C=Date May 1990

C=Block Fortran

```
C -----
C
C Subroutine INPUT
C
C -----
C
C Argument definitions
C
C runtime - ID of analysis run type
C savin - variable to control creation of input file
C runfile - variable indicates if run is from input file
C comment - dummy name for comment input lines
C outskip - number of steps to skip before sending output
C
C subroutine INPUT(runtime,outskip)
C
C include 'shared.inc'
C integer runtime, outskip
C character*1 savin,runfile,temp
C character*48 comment,infile
C
C PRINT AND READ START-UP
C
C print *, '2nd Order Accelerated CSI Simulation (ACSIS)'
C print *
C print *, 'Please input analysis type:'
C print *
C print *, ' 1. Eigenmode Analysis'
C print *, ' 2. CSI Simulation'
C print *, ' 3. Transient Response'
C print *
C read *, runtime
C
C RUN OPTIONS AND INPUT FILE SETUP
C
C runfile = 'n'
C if (runtime .lt. 0) then
C   runtime = -1 * runtime
C   runfile = 'y'
C endif
C print *, 'Do you wish to save an input file? (y or n)'
C read 21, savin
20 format (a32)
21 format (a1)
C if (savin .eq. 'y') then
C   print *, 'Name of save input file? (filename)'
C   read 20, infile
C   open(16,file=infile)
C   runtime = -1 * runtime
C   write(16,'(i2)') runtime
C   runtime = -1 * runtime
C   write(16,'(a1)') 'n'
C   write(16,'(a47)') '* ACSIS input file,two lines above are'
C   write(16,'(a48)') '* analysis type and save input file. Do'
C   write(16,'(a48)') '* not change them by editing this file.'
C endif
```

```

runtype = runtype - 2

if (runfile .eq. 'y') then
  do 30 i = 1,4
    read 20, comment
30    continue
  endif
  print *, 'Finite Element Model Input File Name (filename)'
  read 20, femfile
  open(11,file=femfile)
  if (savin .eq. 'y') then
    write(16,'(a47)') '* Finite element input file?(filename)'
    write(16,'(a32)') femfile
  endif
  if (runtype) 100,200,300

C  EIGENMODE INPUTS

100  print *, 'Number of modes desired:'
  if (runfile .eq. 'y') read 20, comment
  read *, neig
  if (runfile .eq. 'y') read 20, comment
  print *, 'Output File Name:'
  read 20, outfile
  open(13,file=outfile)
  if (savin .eq. 'y') then
    write(16,'(a35)') '* Number of modes desired?'
    write(16,'(i4)') neig
    write(16,'(a33)') '* Output file?(filename)'
    write(16,'(a32)') outfile
  endif

  call READFEM

  goto 1000

C  CSI INPUTS

200  print *, 'Controller Definition File Name:'
  if (runfile .eq. 'y') read 20, comment
  read 20, confile
  open(12,file=confile)
  print *, 'Please input type of control:'
  print *
  print *, ' 1. Full State Feedback'
  print *, ' 2. Luenberger Observer (L1=0)'
  print *, ' 3. Kalman Filter'
  print *
  if (runfile .eq. 'y') read 20, comment
  read *, contype
  contype = contype - 2
  print *, 'Initial time, final time, control-on time, step size:'
  if (runfile .eq. 'y') read 20, comment
  read *, t0,tf,tc,h
  print *, 'Forcing function ID, scale factor, damping coeff- a,b:'
  if (runfile .eq. 'y') read 20, comment
  read *, forceid,scalef,adamp,bdamp
  print *, 'Phase lag fix?(y or n):'
  if (runfile .eq. 'y') read 20, comment

```

```

read 21, temp
if (temp .eq. 'y') nolag = .true.
if (temp .eq. 'n') nolag = .false.
print *, 'Gain scale factors (4 total):'
if (runfile .eq. 'y') read 20, comment
read *, qalpha, qbeta, qalphao, qbetao
if (savin .eq. 'y') then
  write(16, '(a42)') '* Controller file name?(filename)'
  write(16, '(a32)') confile
  write(16, '(a42)') '* Please input type of control: '
  write(16, '(i10)') contype + 2
  write(16, '(a46)') '* Initial, final, control-on, step size?'
  write(16, '(4f14.8)') t0, tf, tc, h
  write(16, '(a49)') '* Forcing function, scale f, damping a, b?'
  write(16, '(i4, f15.6, 2f12.8)') forceid, scalef, adamp, bdamp
  write(16, '(a32)') '* Phase lag fix?(y or n)'
  if (nolag) write(16, '(a1)') 'y'
  if (.not. nolag) write(16, '(a1)') 'n'
  write(16, '(a40)') '* Gain scale factors (4 total)?'
  write(16, '(4f14.8)') qalpha, qbeta, qalphao, qbetao
endif

```

```
call READFEM
```

```
goto 999
```

#### C TRANSIENT RESPONSE INPUTS

```

300 print *, 'Initial time, final time, step size:'
if (runfile .eq. 'y') read 20, comment
read *, t0, tf, h
print *, 'Forcing function ID, scale factor, damping coeff- a, b:'
if (runfile .eq. 'y') read 20, comment
read *, forceid, scalef, adamp, bdamp
if (savin .eq. 'y') then
  write(16, '(a48)') '* Initial, final, step size?'
  write(16, '(3f14.8)') t0, tf, h
  write(16, '(a49)') '* Forcing function, scale f, damping a, b?'
  write(16, '(i4, f15.6, 2f12.8)') forceid, scalef, adamp, bdamp
endif

```

```
call READFEM
```

```
goto 999
```

#### C OUTPUT OPTIONS

```

999 print *, 'Output File Name:'
if (runfile .eq. 'y') read 20, comment
read 20, outfile
open(13, file=outfile)
print *, 'Number of displacement results to output (max 10):'
if (runfile .eq. 'y') read 20, comment
read *, ndisout
do 500 i=1, ndisout
  print *, 'Input node #, dof for displacement output#', i
  read *, outlabel(i), outlabel(i+10)
500 continue
print *, 'Number of velocity results to output (max 10):'

```

```

if (runfile .eq. 'y') read 20, comment
read *,nvelout
do 600 i=1,nvelout
  print *,'Input node $, dof for velocity output$',i
  read *,outlabel(i+20),outlabel(i+30)
600 continue
print *,'Send output every how many steps?'
if (runfile .eq. 'y') read 20, comment
read *, outskip
if (savin .eq. 'y') then
  write(16,'(a38)') '* Output file name?(filename)'
  write(16,'(a32)') outfile
  write(16,'(a42)') '* Number of displacement outputs?'
  write(16,'(i4)') ndisout
  do 650 i=1,ndisout
    write(16,'(2i8)') outlabel(i),outlabel(i+10)
650 continue
  write(16,'(a38)') '* Number of velocity outputs?'
  write(16,'(i4)') nvelout
  do 660 i=1,nvelout
    write(16,'(2i8)') outlabel(i+20),outlabel(i+30)
660 continue
  write(16,'(a44)') '* Send output every how many steps?'
  write(16,'(i3)') outskip
endif

```

#### C ANIMATION OPTION

```

print *,'Animation Output? (y or n):'
if (runfile .eq. 'y') read 20, comment
read 21, temp
if (temp .eq. 'y') animate = .true.
if (temp .eq. 'n') animate = .false.
if (animate) then
  print*,'Animation file name (filename)'
  if (runfile .eq. 'y') read 20, comment
  read 20, animfile
  open (15, file=animfile)
endif
if (savin .eq. 'y') then
  write(16,'(a41)') '* Send animation output?(y or n)'
  if (animate) write(16,'(a1)') 'y'
  if (.not. animate) write(16,'(a1)') 'n'
  if (animate) then
    write(16,'(a31)') '* Animation file name?'
    write(16,'(a32)') animfile
  endif
endif

delta = h/2.
delsq = delta**2

1000 return
end

```

---

File: pmvmul.f

---

C=Module PNVHUL  
C=Author K. Alvin  
C=Date May 1990  
C=Block Fortran

C -----  
C  
C Subroutine PNVHUL  
C  
C Purpose:  
C This subroutine multiplies a matrix in vector form  
C and a vector.  
C

C -----  
C  
C Arguments  
C a - matrix in vector form  
C b - vector  
C c - result vector  
C neq - order of vector and square matrix  
C jdiag - array of diagonal addresses for a  
C

C subroutine PNVHUL(a,jdiag,b,neq,c)  
  
C recursive subroutine PNVHUL(a,jdiag,b,neq,c)  
  
C real\*8 a(1), b(1), c(1)  
C integer jdiag(1), neq  
  
C do 100 i=1,neq  
C     c(i) = a(jdiag(i))\*b(i)  
100 continue  
  
C do 200 i=2,neq  
C     do 300 j=jdiag(i-1)+1,jdiag(i)-1  
C         k = jdiag(i) - j  
C         c(i) = c(i) + a(j)\*b(i-k)  
300     continue  
200 continue  
  
C do 250 i=2,neq  
C     do 400 j=jdiag(i-1)+1,jdiag(i)-1  
C         k = jdiag(i) - j  
C         c(i-k) = c(i-k) + a(j)\*b(i)  
400     continue  
250 continue  
  
C return  
C end

C -----  
C  
C Subroutine PNVHAD  
C  
C Purpose:  
C Multiply a matrix in vector form and a vector and add the  
C resultant vector multiplied by a constant to a second vector  
C



```

C      multiplied by a second vector
C
C -----
C
C      Arguments
C      a      - matrix in vector form
C      b      - vector to be multiplied with matrix
C      c      - resultant and vector to be added
C      fact1  - constant multiplier of matrix and first vector
C      fact2  - constant multiplier of second vector
C      jdiag  - array of diagonal addresses for matrix
C      neq    - order of vectors and matrix
C
C
C      subroutine PHVMAD(a,jdiag,b,neq,fact1,c,fact2)
C
C      recursive subroutine PHVMAD(a,jdiag,b,neq,fact1,c,fact2)
C
C      real*8 a(1), b(1),c(1),fact1,fact2
C      integer jdiag(1), neq
C
C      do 100 i=1,neq
C          c(i) = fact2*c(i) + fact1*a(jdiag(i))*b(i)
100      continue
C
C      do 200 i=2,neq
C          do 300 j=jdiag(i-1)+1,jdiag(i)-1
C              k = jdiag(i) - j
C              c(i) = c(i) + fact1*a(j)*b(i-k)
300      continue
200      continue
C
C      do 250 i=2,neq
C          do 400 j=jdiag(i-1)+1,jdiag(i)-1
C              k = jdiag(i) - j
C              c(i-k) = c(i-k) + fact1*a(j)*b(i)
400      continue
250      continue
C
C      return
C      end

```

---

File: prepfem.f

---

```

C=Module PREPFEM
C=Purpose Preprocess Structure Finite Element module for ACSIS
C=Author K. Alvin
C=Date May 1990
C=Block Fortran
C -----
C
C      Subroutine PREPFEM
C
C      Purpose:
C          This subroutine prepares the finite element mass,
C          stiffness, and S matrices in reduced profile vector form

```

```

C
C -----
C
C Local variables:
C
C   sk      Element Stiffness matrix
C   sm      Element Mass Matrix
C   lm      Local/Global DOF Mapping vector
C   nseq    Number of element degrees of freedom
C   em,ep   Material and Property id # for element
C
C   subroutine PREPFEM
C
C       include 'shared.inc'
C
C LOCAL VARIABLES
C
C   parameter (MAXSEQ=24)
C   real*8 sk(MAXSEQ,MAXSEQ),sm(MAXSEQ,MAXSEQ),mc,ke
C   integer lm(MAXSEQ),nseq,em,ep
C
C   call RENUM
C
C Set up skyline storage profile for global matrices
C
C   call PROFILE(ix,id,jdiag,nnp,nel,4,6,mLen,ndof,mask)
C
C Perform automatic domain decomposition
C
C   call DOMDEC
C
C Check size of skyline profile against storage limitation
C
C   if (mLen .gt. MAXMLEN) then
C     print*, 'PREPFEM: error, global matrix exceeded max. size'
C   endif
C
C Zero Global Matrices prior to assembly
C
C   call ZEROVECT(stif,mLen)
C   call ZEROVECT(mass,mLen)
C
C ASSEMBLE EACH ELEMENT MASS AND STIFFNESS
C
C   do 100 n=1,nel
C
C     do 20 k=1,4
C       j=ix(k,n)
C       if ((etype(n).eq.1).and.(k.gt.2)) j = 0
C       do 30 i=1,6
C         kk=6*(k-1) + i
C         if (j .ne. 0) then
C           lm(kk) = id(i,j)
C         else
C           lm(kk) = 0
C         endif
C       30   continue
C     20   continue

```

```

if (etype(n) .eq. 1) then
  nseq = 12
  em = emat(n)
  ep = eprop(n)
  call BEAM3D(n,ix(1,n),ix(2,n),ix(3,n),coxyz,mat(1,em),
    mat(2,em),mat(3,em),prop(1,ep),prop(5,ep),prop(6,ep),
    prop(2,ep),prop(3,ep),prop(4,ep),pin(1,n),sk,sm)
  elseif (ix(1,n) .ne. 0) then
print*, 'PREPFEM:Element type not found,n=',n,'etype=',etype(n)
endif

```

C ADD ELEMENT TO GLOBAL MASS AND STIFFNESS

```

call ADDSTF(sk,lm,stif,jdiag,nseq)
call ADDSTF(sm,lm,mass,jdiag,nseq)

```

C SAVE THE ELEMENT STIFFNESS FOR E-BY-E COMPUTATIONS

```

call SAVESK(sk,n,nseq)

```

100 continue

C ADD LUMPED INERTIAS TO GLOBAL MASS

```

do 125 i=1,nnp
  do 130 j=1,6
    if (id(j,i) .eq. 0) goto 130
    k=jdiag(id(j,i))
    mass(k) = mass(k) + inertia(j,i)
130   continue
125   continue

```

C ASSEMBLE AND FACTORIZE es (S MATRIX)

```

mc = 1. + delta*adamp
kc = delta*bdamp + delsq
do 200 i=1,mlem
  es(i) = mc*mass(i) + kc*stif(i)
200   continue

```

```

call SOLVER(es,gs,jdiag,ndof,i)

```

C INITIALIZE DISPLACEMENT AND VELOCITY VECTORS

```

do 300 i = 1, nnp
  do 350 j = 1, 6
    if (id(j,i) .ne. 0) then
      q(id(j,i)) = q0(j,i)
      qdot(id(j,i)) = qdot0(j,i)
    endif
350   continue
300   continue

```

```

return
end

```

```

subroutine SAVESK(sk,n,nseq)

```

```

include 'shared.inc'

```

```

real*8 sk(nseq,1)
integer n,nseq

k=0
do 10 j=1,nseq
  do 20 i=1,j
    k=k+1
    estifm(k,n)=sk(i,j)
20    continue
10    continue

return
end

subroutine DOMDEC
include 'shared.inc'

logical nchk,ndchk(MAXNODE,MAXDOM)
integer ndom

do 10 j=1,MAXDOM
  neld(j)=0
  do 20 i=1,nnp
    ndchk(i,j)=.false.
20    continue
10    continue
ndomain=0

do 100 n=1,nel

  ndom=0
  nchk=0
  do 200 while (nchk.eq.0)
    ndom=ndom+1
    if (ndom.gt.ndomain)- ndomain=ndom
    nchk=1
    if (ndchk(ix(1,n),ndom,) nchk=0
    if (ndchk(ix(2,n),ndom)) nchk=0
    if (nchk.eq.1) then
      eldom(n)=ndom
      ndchk(ix(1,n),ndom)=.true.
      ndchk(ix(2,n),ndom)=.true.
    endif
200    continue

    neld(ndom)=neld(ndom)+1
    elnum(neld(ndom),ndom)=n

100    continue

return
end

```

---

File: profile.f

---

C=Module PROFILE

C=Purpose Compute the number of equations and set profile for K

C=Author Bob Taylor

C=Date who knows

C=Update January 1989 by E. Pramono

C=Block Fortran

subroutine PROFILE(ix,id,jdiag,nnp,nel,nen,ndof,nad,neq,mask)

```
C-----+C
C  PURPOSE:                                     C
C  THIS SUBROUTINE COMPUTES THE NUMBER OF EQUATIONS REQUIRED C
C  TO SOLVE THE PROBLEM BY ELIMINATING RESTRAINED DEGREES OF C
C  FREEDOM FROM THE SYSTEM OF EQUATIONS. KNOWING THE EQUATION C
C  NUMBERS COCORRESPONDING TO THE NODAL DEGREES OF FREEDOM, THE C
C  DIAGONAL ELEMENT LOCATIONS CAN BE COMPUTED FOR STORING THE C
C  GLOBAL STIFFNES MATRIX IN COMPACTED VECTOR FORM.       C
C-----C
```

```
C
C  ARGUMENTS
```

```
integer ix(nen,1), id(ndof,1), jdiag(1)
integer nnp, nel, nad, neq, mask(1)
integer nnp, nel, nen, ndof, nad, neq, mask(1)
```

```
C  LOCAL ARGUMENTS
```

```
integer i, j, k, l, m, n, ji, ki, li, mi
```

```
C
C  SET UP EQUATION NUMBERS
```

```
C
C  neq = 0
C  do 30 n = 1, nnp
C    do 20 m = 1, ndof
C      j = id(m,mask(n))
C      if (j .eq. 1) goto 10
C      neq = neq + 1
C      id(m,mask(n)) = neq
C      jdiag(neq) = 0
C      goto 20
10    id(m,mask(n)) = 0
20    continue
30    continue
```

```
C
C  COMPUTE COLUMN HEIGHTS
```

```
C
C  do 80 n = 1, nel
C    do 70 m = 1, nen
C      mi = ix(m,n)
C      if (mi .le. 0) goto 70
C      do 60 l = 1, ndof
C        li = id(l,mi)
C        if (li .eq. 0) goto 60
C        do 50 k = m, nen
C          ki = ix(k,n)
C          if (ki .le. 0) goto 50
C          do 40 j = 1, ndof
C            ji = id(j,ki)
C            if (ji .eq. 0) goto 40
```

```

                i = MAXO(l1,j1)
                jdiag(i) = MAXO(jdiag(i), IABS(l1-j1))
40             continue
50             continue
60             continue
70             continue
80             continue
C
C
C COMPUTE DIAGONAL POINTERS
C
    nad = 1
    jdiag(1) = 1
    if (neq .eq. 1) return
    do 90 n = 2, neq
        jdiag(n) = jdiag(n) + jdiag(n-1) + 1
90     continue
    nad = jdiag(neq)
C
    return
    end
C=End Fortran

```

---

File: read.f

---

```

C=Module READ
C=Author K. Alvin
C=Date May 1990
C=Block Fortran
C -----
C
C Subroutine READFEN
C
C Purpose:
C This subroutine reads the data file for the finite
C element model.
C -----
C
C Arguments
C ctype - stores code for type of line
C
C
C subroutine READFEN
C
C GLOBALS
C
C include 'shared.inc'
C
C LOCALS
C
C integer j,n,ctype,GETTYPE
C character*132 aline
C real*8 in
C
C INITIALIZE SIZE OF PROBLEM

```

```
nnp = 0
nel = 0
ndof = 0
ndomain = 0
```

C IDENTIFY CARD TYPE AND ASSIGN INPUT

```
10 read(11,1000,end=9999) aline
100 ctype = GETTYPE(aline)
    if (ctype) 10,10,150
150 if (aline(1:4) .eq. 'NODE') goto 200
    if (aline(1:4) .eq. 'TOPO') goto 300
    if (aline(1:4) .eq. 'ATTR') goto 400
    if (aline(1:4) .eq. 'MATE') goto 500
    if (aline(1:4) .eq. 'PROP') goto 600
    if (aline(1:4) .eq. 'FIXI') goto 700
    if (aline(1:4) .eq. 'INIT') goto 800
    if (aline(1:4) .eq. 'INER') goto 900
    if (aline(1:4) .eq. 'END ') goto 10
    if (aline(1:4) .eq. 'MESH') goto 10
print *, 'READFEM: Unrecognized card type; ',aline(1:4)
goto 10
```

C READ NODES

```
200 read(11,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 200,250,100
250 read(aline,*) n,(coxyz(j,n),j=1,3)
    if (n .gt. nnp) nnp = n
    goto 200
```

C READ TOPOLOGY

```
300 read(11,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 300,350,100
350 read(aline,*) n,etype(n),(ix(j,n),j=1,4)
    if (n .gt. nel) nel = n
    goto 300
```

C READ ATTRIBUTES

```
400 read(11,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 400,450,100
450 read(aline,*) n,emat(n),eprop(n),(pin(j,n),j=1,6)
    if (eldom(n).gt.ndomain) ndomain=eldom(n)
    goto 400
```

C READ MATERIAL

```
500 read(11,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 500,550,100
550 read(aline,*) n,(mat(j,n),j=1,3)
    goto 500
```

C READ PROPERTIES





```

C      F2GA - control gain matrix
C      L1GA - state estimator filter gain matrix
C      L2GA - state estimator filter gain matrix

```

```

subroutine READCON

```

```

include 'shared.inc'

```

```

C      LOCALS

```

```

real*8 val
integer j,n,ctype,GETTYPE
character*132 aline

```

```

bval = 0
hdval = 0
hvval = 0
hdbval = 0
hvtval = 0

```

```

C      IDENTIFY CARD TYPE AND ASSIGN INPUT

```

```

10  read(12,1000,end=9999) aline
100  ctype = GETTYPE(aline)
    if (ctype) 10,10,150
150  if (aline(1:4) .eq. 'NACT') goto 200
    if (aline(1:4) .eq. 'NSEN') goto 300
    if (aline(1:4) .eq. 'BMAT') goto 400
    if (aline(1:4) .eq. 'HDMA') goto 500
    if (aline(1:4) .eq. 'HVMA') goto 600
    if (aline(1:4) .eq. 'F1GA') goto 700
    if (aline(1:4) .eq. 'F2GA') goto 800
    if (aline(1:4) .eq. 'L2GA') goto 900
    if (aline(1:4) .eq. 'L1GA') goto 1100
    if (aline(1:4) .eq. 'END ') goto 10
    print *, 'READCON: Unrecognized card type; ',aline(1:4)
    goto 10

```

```

C      READ INPUT CARDS

```

```

200  read(12,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 200,250,100
250  read(aline,*) nact
    goto 200

300  read(12,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 300,350,100
350  read(aline,*) nsen
    goto 300

400  read(12,1000,end=9999) aline
    ctype = GETTYPE(aline)
    if (ctype) 400,450,100
450  read(aline,*) i,j,n,val
    bval = bval + 1
    b(bval) = val
    brow(bval) = id(j,i)

```

```

        bcol(bval) = n
        goto 400

500  read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 500,550,100
550  read(aline,*) i,j,n,val
      hdval = hdval + 1
      hd(hdval) = val
      hdrow(hdval) = n
      hdcol(hdval) = id(j,i)
      goto 500

600  read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 600,650,100
650  read(aline,*) i,j,n,val
      hvval = hvval + 1
      hv(hvval) = val
      hvrow(hvval) = n
      hvcol(hvval) = id(j,i)
      goto 600

700  read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 700,750,100
750  read(aline,*) i,j,n,val
      f1(n,id(j,i)) = qalpha*val
      goto 700

800  read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 800,850,100
850  read(aline,*) i,j,n,val
      f2(n,id(j,i)) = qbeta*val
      goto 800

900  read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 900,950,100
950  read(aline,*) i,j,n,val
      l2(id(j,i),n) = qbetao*val
      goto 900

1100 read(12,1000,end=9999) aline
      ctype = GETTYPE(aline)
      if (ctype) 1100,1150,100
1150 read(aline,*) i,j,n,val
      l1(id(j,i),n) = qalphao*val
      goto 1100

1000 format(a132)
9999 continue
      return
      end

```

C  
-----  
C  
C

```

C      Function GETTYPE
C
C      Purpose:
C      This function identifies whether a line is a character
C      input, data input or comment.
C
C      -----
C
C      function GETTYPE(string)
C
C      GLOBALS
C
C      character*132 string
C
C      LOCALS
C
C      integer GETTYPE,ctype(10)
C      character*1 head(10)
C
C      data head /'!', '@', '#', '$', '%', '&', '*', 'C', 'c', ' ' /
C      data ctype /-1,-1,-1,-1,-1,-1,-1,-1,-1,0/
C
C      LOGIC
C
C      GETTYPE=1
C      do 100 i=1,10
C          if (string(1:i) .eq. head(i)) GETTYPE=ctype(i)
100      continue
C
C      return
C      end

```

---

File: solver.f

---

```

C=Module SOLVER
C=Purpose Solves the system of linear symmetric equations
C=Author who knows
C=Date
C=Update January 1989 by E. Pramono
C=Block Fortran
C      SUBROUTINE SOLVER(BK, BR, JDIAG, NEQ, IFLAG)
C      recursive SUBROUTINE SOLVER(BK, BR, JDIAG, NEQ, IFLAG)
C+-----+C
C      PURPOSE:
C      THIS SUBROUTINE SOLVES THE SYSTEM OF LINEAR SYMMETRIC
C      EQUATIONS IN VECTOR FORM USING THE CROUT REDUCTION
C      METHOD.
C
C      ARGUMENTS:
C      BK      - GLOBAL STIFFNESS EQUATIONS IN VECTOR FORM
C      BR      - GLOBAL LOAD VECTOR
C      JDIAG   - LOCATION VECTOR FOR DIAGONALS IN [BK]
C      NEQ     - NUMBER OF EQUATIONS
C      IFLAG   - FLAG INDICATING WHICH FUNCTION IS TO BE PERFORMED
C                1 -> FORWARD REDUCTION
C                2 -> BACKWARD SUBSTITUTION

```

```

-----C
C ARGUMENTS

REAL*8 BK(1), BR(1)
INTEGER JDIAG(1), NEQ, IFLAG

C LOCAL VARIABLES
REAL*8 ZERO, EZERO, TOL, DAVAL, DOT, D, RDD, DD
INTEGER LDFLAG, JR, J, JD, JH, IS, IE, K, JDT
INTEGER JJ, ID, I, IR, IH

JJ = 6

C
C -----
C NEW PARAMETERS
C -----
ZERO = 0.000
EZERO = 0.3D-14
TOL = 0.5D-7
LDFLAG = 0

C
C -----
C FACTOR BK TO UT*D*U OR REDUCE R
C -----
JR = 0
DO 70 J = 1, NEQ
JD = JDIAG(J)
JH = JD - JR
IS = J - JH + 2
IF (JH - 2) 60, 30, 10

C
10 IF (IFLAG .NE. 1) GOTO 50
IE = J - 1
K = JR + 2
ID = JDIAG(IS-1)

C
C -----
C IF DIAGONAL IS ZERO COMPUTE A NORM FOR SINGULARITY TEST
C -----
JDT = JDIAG(IE) + 1
IF (BK(JD) .EQ. ZERO .AND. IFLAG .EQ. 1) THEN
CALL DATEST (BK(JDT), JH-2, DAVAL)
END IF

C
C -----
C REDUCE ALL EQUATIONS EXCEPT FIRST ROW AND DIAGONAL
C -----
DO 20 I = IS, IE
IR = ID
ID = JDIAG(I)
IH = MINO(ID-IR-1, I-IS+1)
IF (IH .GT. 0) BK(K) = BK(K) - DOT(BK(K-IH), BK(ID-IH), IH)
K = K + 1
20 CONTINUE

C
C -----
C REDUCE FIRST ROW AND DIAGONAL

```

```

C -----
30  IF (IFLAG .NE. 1) GOTO 50
    IR = JR + 1
    IE = JD - 1
    K = J - JD
    DD = BK(JD)
    DO 40 I = IR, IE
        ID = JDIAG(K+I)
        IF (BK(ID) .EQ. 0.0) GOTO 40
        D = BK(I)
        BK(I) = BK(I)/BK(ID)
        BK(JD) = BK(JD) - D*BK(I)
40  CONTINUE
C
C -----
C  CHECK FOR POSSIBLE ERRORS AND PRINT WARNINGS
C -----
    RDD = BK(JD)
    IF (DABS(RDD) .LT. TOL*DABS(DD)) WRITE (JJ,2000) J
    IF (DD .LT. ZERO .AND. RDD .GT. ZERO) WRITE (JJ,2001) J
    IF (DD .GT. ZERO .AND. RDD .LT. ZERO) WRITE (JJ,2001) J
    IF (DABS(RDD) .LT. EZERO) WRITE (JJ,2002) J
C
C -----
C  COMPLETE RANK TEST FOR A ZERO DIAGONAL TEST
C -----
    IF (DD .EQ. ZERO .AND. JH .GT. 0) THEN
        IF (DABS(RDD) .LT. TOL*DAVAL) WRITE (JJ,2003) J
    END IF
C
C -----
C  REDUCE RIGHT HAND SIDE
C -----
50  IF (IFLAG .EQ. 2) BR(J) = BR(J) - DOT(BK(JR+1), BR(IS-1), JH-1)
60  JR = JD
70  CONTINUE
    IF (IFLAG .NE. 2) RETURN.
C
C -----
C  DIVIDE BY DIAGONAL TERMS
C -----
    DO 80 I = 1, NEQ
        ID = JDIAG(I)
        IF (BK(ID) .NE. 0.0) BR(I) = BR(I)/BK(ID)
        IF (BR(I) .NE. ZERO) LDFLAG = 1
80  CONTINUE
C
C -----
C  CHECK FOR ZERO LOAD VECTOR
C -----
    IF (LDFLAG .EQ. 0) WRITE(JJ,2004)
C
C -----
C  BACK SUBSTITUTE
C -----
    J = NEQ
    JD = JDIAG(J)
90  D = BR(J)
    J = J - 1

```

```

IF (J .LE. 0) RETURN
JR = JDIAG(J)
IF (JD - JR .LE. 1) GOTO 110
IS = J - JD + JR + 2
K = JR - IS + 1
DO 100 I= IS, J
    BR(I) = BR(I) - BK(I+K)*D
100 CONTINUE
110 JD = JR
GOTO 90

```

```

C
C -----
C WARNING FORMATS
C -----

```

```

2000 FORMAT(/'!! WARNING !! 1 - IN SOLVER, LOSS OF AT LEAST 7 DIGITS'
+ /18X, 'IN REDUCING DIAGONAL OF EQUATION;',4X,I5)
2001 FORMAT(/'!! WARNING !! 2 - IN SOLVER, SIGN OF DIAGONAL CHANGED'
+ /18X, 'WHEN REDUCING EQUATION;',15X,I5)
2002 FORMAT(/'!! WARNING !! 3 - IN SOLVER, REDUCED DIAGONAL IS ZERO'
+ /18X, 'FOR EQUATION;',25X,I5)
2003 FORMAT(/'!! WARNING !! 4 - IN SOLVER, RANK FAILURE FOR A ZERO'
+ /18X, 'UNREDUCED DIAGONAL IN EQUATION;',7X,I5)
2004 FORMAT(/'!! WARNING !! 5 - IN SOLVER, ZERO LOAD VECTOR')

```

```

C
END

```

```

C=End Fortran
C=Module DATEST
C=Block Fortran

```

```

C SUBROUTINE DATEST(A,JH,DAVAL)
recursive SUBROUTINE DATEST(A,JH,DAVAL)

```

```

C+-----+C
C TEST FOR RANK C
C INPUTS; C
C A(J) - COLUMN OF UNREDUCED ELEMENTS IN ARRAY C
C JH - NUMBER OF ELEMENTS IN COLUMN C
C OUTPUTS; C
C DAVAL - SUM OF ABSOLUTE VALUES C
C+-----+C

```

```

C ARGUMENTS

```

```

REAL*8 A(1), DAVAL
INTEGER JH

```

```

C LOCAL ARGUMENTS

```

```

INTEGER J
C
DAVAL = 0.0D0
DO 10 J = 1, JH
    DAVAL = DAVAL + DABS(A(J))
10 CONTINUE
C
RETURN
END

```

```

C
C
C=End Fortran
C=Module DOT
C=Block Fortran
C   FUNCTION DOT(A,B,N)
C     recursive FUNCTION DOT(A,B,N)
C+-----+C
C   PURPOSE:                                     C
C     THIS FUNCTION SUBROUTINE PERFORMS THE DOT PRODUCT OF TWO   C
C     VECTORS.                                         C
C                                                     C
C   ARGUMENTS:                                       C
C     A  - FIRST VECTOR INVOLVED IN DOT PRODUCT           C
C     B  - SECOND VECTOR INVOLVED IN DOT PRODUCT          C
C     N  - NUMBER OF ELEMENTS IN EACH OF THE TWO VECTORS  C
C-----C
C
C     REAL*8 DOT, A(1), B(1)
C     INTEGER N
C
C     INTEGER I
C
C     DOT = 0.0
C     DO 10 I = 1, N
C       DOT = DOT + A(I)*B(I)
10    CONTINUE
C
C     RETURN
C     END
C=End Fortran

```

File: nophlag.f

```

C=Module NOPHLAG
C=Author K. Alvin
C=Date July 1990
C=Block Fortran
C-----C
C   Subroutine NOPHLAG
C
C   Purpose:
C     This subroutine solves for the structural displacement
C     and velocity vectors at the half-step for the phase lag
C     correction loop, and gets new measurement
C-----C
C   Arguments
C     delbeta - delta * bdamp
C
C
C   subroutine NOPHLAG(zp)
C
C   real*8 zp(1)
C   include 'shared.inc'

```

```

C   LOCAL VARIABLES

integer i
real*8 v(MAXDOF),delbeta

C   LOGIC
C   ADD APPLIED FORCES TO RHS AND PREPARE MASS MULTIPLIER

do 10 i=1,ndof
  gs(i) = gs(i) + f(i)
  v(i) = (1. + delta*adamp)*q(i) + delta*qdot(i)
10  continue

C   SOLVE FOR RIGHT HAND SIDE, gs

do 77 i = 1,ndof
  gs(i) = delsq*gs(i) + v(i)*mass(jdiag(i))
77  continue

  if (bdamp .ne. 0.) then
    delbeta = delta*bdamp
    call PNVHAD(stif,jdiag,q,ndof,delbeta,gs,1.d0)
  endif

C   SOLVE FOR DISPLACEMENT, q, USING RHS AND MATRIX S

call SOLVER(es,gs,jdiag,ndof,2)

do 100 i=1,ndof
  v(i) = (gs(i)-q(i))/delta
100 continue

call ZEROVECT(zp,nzen)

do 200 jj = 1,hdval
  i = hdrow(jj)
  j = hdcol(jj)
  zp(i) = zp(i) + hd(jj)*gs(j)
200 continue

do 250 jj = 1,hvval
  i = hvrow(jj)
  j = hvcol(jj)
  zp(i) = zp(i) + hv(jj)*v(j)
250 continue

return
end

```

---

File: zerovect.f

---

C=Module ZEROVECT  
C=Purpose Initialize vector of given length to zero  
C=Author K. Alvin  
C=Date May 1990  
C -----



```

C
C   Subroutine ZEROVECT
C
C -----
C
C
C   subroutine ZEROVECT(v,n)
C
C   real*8 v(1)
C   integer n
C
C   do 100 i=1,n
C       v(i) = 0.d0
100   continue
C
C   return
C   end

```

File: lu.f

```

SUBROUTINE LUFACT(A,N,PIVOT,DET,IER,NMAX)
C*****
C
C   SUBROUTINE FACTOR USES GAUSSIAN ELIMINATION WITH
C   PARTIAL PIVOTING AND IMPLICIT SCALING TO DETERMINE
C   THE L+U DECOMPOSITION OF A SQUARE MATRIX "A" OF
C   ORDER N. THE ALGORITHM ALSO FINDS THE DETERMINENT
C   OF "A". UPON COMPLETION, THE ELEMENTS OF THE UPPER
C   TRIANGULAR MATRIX "U" ARE CONTAINED IN THEIR RESPECTIVE
C   LOCATIONS IN MATRIX "A". THE ELEMENTS OF MATRIX "L"
C   ARE CONTAINED IN THE LOWER TRIANGULAR PORTION OF "A",
C   BUT ARE SCRAMBLED WITH RESPECT TO "U" BECAUSE OF ROW
C   INTERCHANGE OPERATIONS NOT PERFORMED ON THE ELEMENTS
C   OF "L". THE VECTOR PIVOT (SEE BELOW) MUST BE USED TO
C   UNSCRAMBLE "L" IF IT IS TO BE USED FOR OTHER OPERATIONS.
C
C   VARIABLES:  A=FULL SQUARE MATRIX (DOUBLE PRECISION)
C               N=ORDER OF MATRIX A (INTEGER)
C               PIVOT=VECTOR CONTAINING A RECORD OF
C                   ROW INTERCHANGES. THE INTEGER
C                   VALUE PIVOT(K) IS THE ROW WHICH
C                   WAS INTERCHANGED WITH ROW K AT
C                   FORWARD ELIMINATION STEP K. (INTEGER)
C               DET=DETERMINENT OF MATRIX A (DOUBLE PRECISION)
C               IER=ERROR FLAG. IF IER=1, THE MATRIX A WAS FOUND
C                   TO BE SINGULAR, AND THE ROUTINE WAS EXITED. IF
C                   IER=0, THE DECOMPOSITION WAS SUCCESSFUL.
C
C*****
C   INTEGER PIVOT(1),IER,N,I,J,K,IO,NMAX
C   REAL*8 A(NMAX,1),S(1000),C(1000),DET,TEMP
C   DET=1.0D0
C*****
C
C   FIND THE ROW NORMALIZING COEFFICIENTS S(I) FOR IMPLICIT SCALING.
C   EXIT ROUTINE IF ANY S(I)=0.0

```

```

C
C*****
DO 100 I=1,N
  S(I)=0.000
  DO 110 J=1,N
    IF (ABS(A(I,J)).GT.S(I)) S(I)=ABS(A(I,J))
110  CONTINUE
    IF (S(I).EQ.0) THEN
      IER=1
      DET=0.000
      RETURN
    END IF
100  CONTINUE
C*****
C
C      START FORWARD ELIMINATION STEP K
C
C*****
DO 120 K=1,N-1
C*****
C
C      DETERMINE PIVOT ELEMENT A(IO,K) BY FINDING THE ROW IO
C      BETWEEN K AND N CONTAINING THE MAXIMUM NORMALIZED
C      VALUE IN COLUMN K. SET PIVOT(K)=IO
C
C*****
C(K)=0.000
DO 130 I=K,N
  TEMP=ABS(A(I,K)/S(I))
  IF (TEMP.GT.C(K)) THEN
    C(K)=TEMP
    IO=I
  END IF
130  CONTINUE
  PIVOT(K)=IO
C*****
C
C      EXIT ROUTINE IF ALL VALUES IN COLUMN K AT OR BELOW
C      THE MAIN DIAGONAL ARE EQUAL TO 0.0
C
C*****
IF (C(K).EQ.0.0) THEN
  IER=1
  DET=0.000
  RETURN
END IF
C*****
C
C      INTERCHANGE ROWS IO AND K FOR COLUMNS K TO N. SKIP IF IO=K.
C      SET DET=-DET IF ROWS ARE INTERCHANGED.
C
C*****
IF (IO.EQ.K) GOTO 150
DET=-1.000*DET
DO 140 J=K,N
  TEMP=A(K,J)
  A(K,J)=A(IO,J)
  A(IO,J)=TEMP
140  CONTINUE

```

```

C*****
C
C   ELIMINATE COLUMN K BELOW MAIN DIAGONAL BY MULTIPLYING
C   ROW K FROM COLUMN K TO N BY A(I,K)/A(K,K) AND SUBTRACTING
C   FROM ROW I. STORE THE MULTIPLIER FOR ROW I IN THE ELIMINATED
C   COLUMN K. MULTIPLY THE RUNNING PRODUCT DET BY DIAGONAL ELEMENT A(K,K).
C

```

```

C*****
150  DO 160 I=K+1,N
      A(I,K)=A(I,K)/A(K,K)
      DO 170 J=K+1,N
        A(I,J)=A(I,J)-A(I,K)*A(K,J)
170   CONTINUE
160   CONTINUE
      DET=DET*A(K,K)
120  CONTINUE

```

```

C*****
C
C   CHECK LAST ROW/COLUMN FOR SINGULARITY. IF THERE IS NO ERROR,
C   COMPLETE CALCULATION OF THE DETERMINANT, SET THE ERROR FLAG
C   TO INDICATE NORMAL COMPLETION, AND EXIT.
C

```

```

C*****
      IF (A(N,N).EQ.0.0) THEN
        IER=1
        DET=0.0D0
        RETURN
      END IF
      DET=DET*A(N,N)
      IER=0
      RETURN
    END

```

SUBROUTINE LUSOLVE(A,N,B,PIVOT,NMAX)

```

C*****
C
C           SUBROUTINE SOLVE
C

```

```

C*****
      INTEGER PIVOT(1),N,I,J,K,NMAX
      REAL*8 A(NMAX,1),B(1),TEMP
      DO 100 K=1,N-1
        IF (PIVOT(K).EQ.K) GOTO 110
        TEMP=B(K)
        B(K)=B(J)
        B(J)=TEMP
110   DO 120 I=K+1,N
        B(I)=B(I)-A(I,K)*B(K)
120   CONTINUE
100   CONTINUE
      B(N)=B(N)/A(N,N)
      DO 130 I=N-1,1,-1
        DO 140 J=I+1,N
          B(I)=B(I)-A(I,J)*B(J)
140   CONTINUE
        B(I)=B(I)/A(I,I)
130   CONTINUE
      RETURN
    END

```

```

C=END FORTRAN
c=DECK FACTA
c=PURPOSE - Factors the A matrix as L U = A, with partial pivoting
c=AUTHOR W K BELVIN, Sept. 24, 1987
c
c -----
c   Input
c   amat----[n X n] matrix to be factored, destroyed on output
c   np-----problem size
c
c   Output
c   amat----contains the LU decomposition
c
c -----
c   subroutine FACTA(amat,np,nrow,lp)
c
c   real*8 amat(*),eta
c   integer lp(*)
c
c   do 50 i=1,np
50  lp(i)=i
c
c   Find largest pivot.
c
c   do 100 k=1,np-1
c   amax=0.
c   mmax=0
c   do 200 m=k,np
c   if (abs(amat(lp(m)+(k-1)*nrow)) .gt. amax) then
c   amax=abs(amat(lp(m)+(k-1)*nrow))
c   mmax=m
c   endif
200  continue
c
c   l=lp(k)
c   lp(k)=lp(mmax)
c   lp(mmax)=l
c
c   do 400 i=k+1,np
c   eta=amat(lp(i)+(k-1)*nrow)/amat(lp(k)+(k-1)*nrow)
c   amat(lp(i)+(k-1)*nrow)=eta
c   do 500 j=k+1,np
c   amat(lp(i)+(j-1)*nrow)=amat(lp(i)+(j-1)*nrow)-
c   eta*amat(lp(k)+(j-1)*nrow)
500  continue
400  continue
c
c   100 continue
c
c   return
c   end

C END FORTRAN
c=DECK LUSOLV
c=PURPOSE - Solve L U x = b,
c=AUTHOR W K BELVIN, Sept. 24, 1987
c
c -----

```

```

c   First solves  $L y = b$ , then  $U x = y$ 
c
c   Input
c       amat----[n X n] matrix factored by FACTA
c       np-----problem size
c       lp      --pointer vector based on pivoting
c       rhs----RHS of equation
c
c   Output
c       amat----contains the LU decomposition
c       x-----the solution vector
c
c -----
c   subroutine LUSOLV(amat,np,nrow,lp,rhs,x)
c
c       real*8 amat(*),x(*),rhs(*)
c       integer lp(*)
c
c       do 50 i=1,np
c           x(i)=rhs(i)
c   50  continue
c
c   Solve Lower System-----
c**** Outer loop
c
c       do 100 k=1,np
c           rhs(k)=x(lp(k))
c           if (rhs(k) .eq. 0.) go to 100
c
c**** Inner loop
c
c       do 200 i=k+1,np
c           x(lp(i))=x(lp(i))-amat(lp(i)+(k-1)*nrow)*rhs(k)
c   200  continue
c   100  continue
c
c   Solve Upper System-----
c
c       do 300 k=np,1,-1
c           x(k)=rhs(k)/amat(lp(k)+(k-1)*nrow)
c           do 400 i=1,k-1
c               rhs(i)=rhs(i)-x(k)*amat(lp(i)+(k-1)*nrow)
c   400  continue
c   300  continue
c       return
c       end

```

---

File: prepcon.f

---

```

C=Module PREPCON
C=Purpose Preprocess control analysis module for ACSIS
C=Author K. Alvin
C=Date June 1990
C=Block Fortran
C -----
C
C   Subroutine PREPCON

```

```

C
C   Purpose:
C       This subroutine prepares ec, the control prediction integration
C       matrix and eo, the observer construct matrix S ( $M+\delta*D+delsq*K$ )
C
C -----
C
C
C       subroutine PREPCON
C
C       include 'shared.inc'
C
C   LOCAL VARIABLES
C
C       real*8 mc, kc
C       integer ier
C
C   LOGIC
C
C       call READCON
C
C   Form Control Prediction Integration Coefficient Matrix
C
C       contype = -1 : Full State Feedback
C       contype = 0  : Luenberger Observer
C       contype = +1 : Kalman Filter
C
C       do 10 i = 1, nact + nsen
C         do 20 j = 1, nact + nsen
C           ec(i, j) = 0.d0
20          continue
10          continue
C
C       if (contype) 100, 200, 400
C
100      ncsi = nact
C
C       do 110 i = 1, nact
C         do 120 jj = 1, bval
C           j = bcol(jj)
C           k = brow(jj)
C           ec(i, j) = ec(i, j) + delta*f2(i, k)*b(jj)/mass(jdiag(k))
120          continue
110          continue
C
C       goto 600
C
200      ncsi = nact + nsen
C
C       do 210 i = 1, nact
C         do 220 jj = 1, bval
C           j = bcol(jj)
C           k = brow(jj)
C           ec(i, j) = ec(i, j) + delta*f2(i, k)*b(jj)/mass(jdiag(k))
220          continue
C         do 240 j = nact+1, ncsi
C           do 250 k = 1, ndof
C             ec(i, j) = ec(i, j) + delta*f2(i, k)*l2(k, j-nact)

```

```

250     continue
240     continue
210     continue
do 260 ii = 1,hvval
  i = nact + hvrow(ii)
do 270 jj = 1,bval
  j = bcol(jj)
  k = brow(jj)
  if (hvcoll(ii) .ne. k) goto 270
  ec(i,j) = ec(i,j) + delta*hv(ii)*b(jj)/mass(jdiag(k))
270     continue
do 290 j = nact+1,ncsi
  k = hvcoll(ii)
  ec(i,j) = ec(i,j) + delta*hv(ii)*l2(k,j-nact)
290     continue
280     continue

goto 600

400 ncsi = nact + nsen

do 470 i = 1,nact
do 480 jj = 1,bval
  j = bcol(jj)
  k = brow(jj)
  ec(i,j) = ec(i,j) + delta*f2(i,k)*b(jj)/mass(jdiag(k))
480     continue
do 500 j = nact+1,ncsi
do 510 k = 1,ndof
  ec(i,j) = ec(i,j) + delta*f2(i,k)*l2(k,j-nact)/
    mass(jdiag(k))
510     continue
500     continue
470     continue
do 520 ii = 1,hvval
  i = nact + hvrow(ii)
do 530 jj = 1,bval
  j = bcol(jj)
  k = brow(jj)
  if (hvcoll(ii) .ne. k) goto 530
  ec(i,j) = ec(i,j) + delta*hv(ii)*b(jj)/mass(jdiag(k))
530     continue
do 550 j = nact+1,ncsi
  k = hvcoll(ii)
  ec(i,j) = ec(i,j) + delta*hv(ii)*l2(k,j-nact)/
    mass(jdiag(k))
550     continue
520     continue

600     continue

do 1100 i = 1,ncsi
  ec(i,i) = ec(i,i) + 1.d0
1100     continue

C     FACTORIZE ec

call FACTA(ec,ncsi,MAXCSI,pivot)

```

```

if (ier .eq. 1) then
  print *, 'PREPCON: Singular Matrix for Control Integration'
endif

C Form Observer Integration Coefficient Matrix, eo

mc = 1. + delta*edamp
kc = delta*bdamp + delsq
do 1200 i=1,mlen
  eo(i) = mc*mass(i) + kc*stif(i)
1200 continue

C FACTORIZE eo

call SOLVER(eo,go,jdiag,ndof,1)

C Initialize Observer States

call ZEROVECT(qe,ndof)
call ZEROVECT(qedot,ndof)
call ZEROVECT(pe,ndof)

return
end

```

---

File: control.f

---

```

C=Module CONTROL
C=Author K. Alvin
C=Date May 1990
C=Block Fortran
C -----
C
C Subroutine CONTROL
C
C Purpose:
C This subroutine carries out the numeric integration of one time
C step of the control system.
C -----
C
C Arguments
C qep - estimated displacement vector at half time step
C qedotp - estimated velocity vector at half time step
C pp - generalized momentum (f-D*qedotp-K*qep)
C z - measured sensor output
C
C subroutine CONTROL(z)
C
C include 'shared.inc'
C real*8 z(1)
C
C LOCAL VARIABLES
C
C real*8 qep(MAXDOF),qedotp(MAXDOF),pp(MAXDOF),v(MAXDOF)

```



```

C      LOGIC

C      Form RHS of Control Prediction Equation Set
C
C      contype = -1 : Full State Feedback
C      contype = 0 : Luenberger Observer with  $L_1 = 0$ 
C      contype = +1 : Kalman Filter w/generalized momentum variable

      call ZEROVECT(gc,ncsi)

      if (contype) 100,200,300

100    continue

      do 110 i = 1,ndof
         qe(i) = q(i)
         qedot(i) = qedot(i)
110    continue

200    continue

      do 210 i = 1,ndof
         qep(i) = qe(i) + delta*qedot(i)
         qedotp(i) = qedot(i)
         pp(i) = f(i) - mass(jdiag(i))*adamp*qedotp(i)
         v(i) = qep(i) + bdamp*qedotp(i)
210    continue

      call PNMHAD(stif,jdiag,v,ndof,-1.d0,pp,1.d0)

      do 220 i=1,nact
         do 230 j = 1,ndof
            gc(i) = gc(i) - f1(i,j)*qep(j) - f2(i,j)*(qedot(j) +
              delta*pp(j)/mass(jdiag(j)))
230        continue
220    continue

      if (ncsi .eq. nact) goto 600

      do 240 i=nact+1,ncsi
         k = i - nact
         gc(i) = z(k)
240    continue

      do 245 ii = 1,hdval
         i = hrow(ii) + nact
         j = hcol(ii)
         gc(i) = gc(i) - hd(ii)*qep(j)
245    continue

      do 250 ii = 1,hvval
         i = hrow(ii) + nact
         j = hcol(ii)
         gc(i) = gc(i) - hv(ii)*(qedot(j)+delta*pp(j)/mass(jdiag(j)))
250    continue

      goto 600

300    continue

      do 310 i = 1,ndof

```

```

    qep(i) = qe(i)
    pp(i) = f(i) - mass(jdiag(i))*adamp*qep(i)/delta
    v(i) = (i + bdamp/delta)*qep(i)
310    continue

    call PNVNAD(stif,jdiag,v,ndof,-1.d0,pp,1.d0)

    do 320 i=1,nact
      do 330 j = 1,ndof
        gc(i) = gc(i) - f1(i,j)*qep(j) - f2(i,j)*(pe(j) +
          delta*pp(j))/mass(jdiag(j))
330      continue
320    continue
    do 340 i=nact+1,ncsi
      k = i - nact
      gc(i) = z(k)
340    continue
    do 345 ii = 1,hdval
      i = hdrow(ii) + nact
      j = hdcol(ii)
      gc(i) = gc(i) - hd(ii)*qep(j)
345    continue
    do 350 ii = 1,hvval
      i = hvrow(ii) + nact
      j = hvcol(ii)
      gc(i) = gc(i) - hv(ii)*(pe(j) + delta*pp(j))/mass(jdiag(j))
350    continue

C    FIND r, CONTROL AND STATE CORRECTION FORCES

600    call LUSOLV(ec,ncsi,MAXCSI,pivot,gc,r)

    do 610 j=1,nact
      u(j) = r(j)
610    continue
    do 620 j=nact+1,ncsi
      gamma(j-nact) = r(j)
620    continue

C    FIND CONTROL CONTRIBUTION TO RHS VECTOR FOR
C    OBSERVER AND STRUCTRE

    do 710 i=1,ndof
      gs(i) = 0.d0
      go(i) = 0.d0
      gk(i) = 0.d0
710    continue
    do 720 jj=1,bval
      i = brow(jj)
      j = bcol(jj)
      gs(i) = gs(i) + b(jj)*u(j)
      go(i) = go(i) + b(jj)*u(j)
      gk(i) = gk(i) + b(jj)*u(j)
720    continue
    if (contype .eq. 0) then
      do 725 i = 1,ndof
        do 730 j=1,nsen
          go(i) = go(i) + mass(jdiag(i))*l2(i,j)*gamma(j)
730        continue

```

```

725     continue
      elseif (contype .eq. 1) then
        do 735 i = 1,ndof
          do 740 j=1,nseq
            go(i) = go(i) + (l2(i,j)+mass(jdiag(i))*l1(i,j)/delta)
                          *gamma(j)
            gk(i) = gk(i) + l2(i,j)*gamma(j)
740     continue
735     continue
      endif

      return
      end

```

---

File: recorder.f

---

C=Module RECORDER  
C=Author K. Alvin  
C=Date May 1990  
C=Block Fortran

```

C -----
C
C   Subroutine RECORDER
C
C   Purpose:
C
C   Solves the second-order dynamical equation:
C
C        $Mx'' + Dx' + Kx = f + g$ 
C
C   at time (n+1) given f(n+1/2), g(n+1/2) and x,x' at n by
C   the midpoint implicit integration rule.
C   Step size is 2*delta.
C
C   D is of the form (alpha*M + beta*K), f is an applied force,
C   and g is assumed to be other applied force from a feedback
C   control loop. The matrix E is the factored form of the
C   integration coefficient matrix: E=(M + delta*D + delta^2*K).
C -----
C
C   Arguments:
C
C   m       - matrix M
C   k       - matrix K
C   alpha   - scalar alpha
C   beta    - scalar beta
C   f       - Force vector f(n+1/2)
C   g       - Feedback force vector g(n+1/2)
C   e       - matrix E
C   x       - Variable vector x(n)
C   xd      - Variable vector x'(n)
C   delta   - Half of integration time step
C   delsq   - delta^2
C   jdiag   - Diagonal location pointer for M,K,E matrices
C   ndof    - Number of equations and length of q

```

```

C      v      - mass multiplier for RHS preparation
C      delbeta - delta * beta

C      subroutine SECCORDER(m,k,alpha,beta,f,g,e,x,xd,
C      .      delta,delsq,jdiag,ndof,MAXDOF)

      recursive subroutine SECCORDER(m,k,alpha,beta,f,g,e,x,xd,
      .      delta,delsq,jdiag,ndof,MAXDOF)

C      ARGUMENTS

      real*8 m(1),k(1),alpha,beta,f(1),g(1),e(1)
      real*8 x(1),xd(1),delta,delsq
      integer jdiag(1),ndof,MAXDOF

C      LOCAL VARIABLES

      integer i
      real*8 v(3000),delbeta

C      LOGIC
C      ADD APPLIED FORCES TO RHS AND PREPARE MASS MULTIPLIER

      do 10 i=1,ndof
         g(i) = g(i) + f(i)
         v(i) = (1. + delta*alpha)*x(i) + delta*xd(i)
10      continue

C      SOLVE FOR RIGHT HAND SIDE, g

      do 77 i=1,ndof
         g(i) = delsq*g(i) + v(i)*m(jdiag(i))
77      continue

      if (beta .ne. 0.) then
         delbeta = delta*beta

C      Activate EBE computations for internal force by using STIFFRC
C      subroutine. Otherwise use PHVMAD (profile matrix/vector mult-add)

C      call PHVMAD(k,jdiag,x,ndof,delbeta,g,1.d0)
      call STIFFRC(x,delbeta,g)
      endif

C      SOLVE FOR DISPLACEMENT, q, USING RHS AND MATRIX E

      call SOLVER(e,g,jdiag,ndof,2)

      do 100 i=1,ndof
         xd(i) = 2.*(g(i) - x(i))/delta - xd(i)
         x(i) = 2.*g(i) - x(i)
100      continue

      return
      end

```

File: measure.f

---

C=Module MEASURE  
C=Author K. Alvin  
C=Date May 1990  
C=Block Fortran

C -----  
C  
C Subroutine MEASURE  
C  
C Purpose:  
C This subroutine stores new measured sensor data by using the  
C previous displacement and velocity vectors at the sensor  
C locations  
C -----  
C  
C Arguments  
C zp - measured sensor data array  
C

```
subroutine MEASURE(zp)
include 'shared.inc'
real*8 zp(1)

call ZEROVECT(zp,nsen)

do 100 jj = 1,hdval
  i = hdrow(jj)
  j = hdc col(jj)
  zp(i) = zp(i) + hd(jj)*q(j)
100 continue
do 200 jj = 1,hvval
  i = hvrow(jj)
  j = hvcol(jj)
  zp(i) = zp(i) + hv(jj)*qdot(j)
200 continue

return
end
```

---

File: eigens.f

---

C=Module EIGENS  
C=Purpose Find Eigenmodes given Mass, Stiffness Matrices  
C=Author K. Alvin  
C=Date March 1990  
C=Block Fortran

C -----  
C  
C subroutine EIGENS  
C -----  
C  
C COMMON AND GLOBALS

```
include 'shared.inc'
```

C LOCAL VARIABLES

```
parameter(NVM=100,MNNVM=MAXDOF*NVM,MXCNV=NVM*(NVM+1)/2)
integer isl(MAXDOF),nvec,i,j,k,kk,out
real*8 vl(MNNVM),vr(MNNVM),akk(MXCNV),amm(MXCNV)
real*8 xx(NVM,NVM),eigv(NVM),eigold(NVM)
real*8 toleig,toljac,omega,fhz
```

```
toleig=.0001
out = 13
```

```
toljac = toleig
nvec = min0(2*neig,100)
nvec = min0(nvec,ndof)
```

C SET UP ISL VECTOR

```
isl(1) = 1
do 50 j=2,ndof
  isl(j) = j - jdiag(j) + jdiag(j-1) + 1
50 continue
```

C CALL EIGENSOLVER

```
call SSPACE(stif,mass,vl,vr,akk,amm,xx,eigv,eigold,isl,
  jdiag,neig,nvec,ndof,toleig,toljac,out)
```

C WRITE OUTPUT

```
write(out,*) 'EIGEN ANALYSIS RESULTS:'
write(out,*) '
write(out,*) ' MODE          EIGENVALUE          RADIAL          CYCLIC'
write(out,*) ' ----          -' write(out,*) ' ----          -'
write(out,*) '
do 100 i=1,neig
  omega = dsqrt(eigv(i))
  fhz = omega/(2*3.141592654)
  write(out,'(i5,3(3x,g12.5))') i,eigv(i),omega,fhz
100 continue

write(out,*) 'EIGENVECTORS:'
do 200 j=1,neig,5
  write(out,*)
  do 300 i=1,ndof
    k = ndof*(j-1) + i
    write(out,'(i5,5(1x,g12.5))') i,(vr(kk),kk=k,k+4*ndof,ndof)
300 continue
200 continue

write(out,*) 'MASS MATRIX DIAGONAL:'
do 400 i=1,nnp
  do 450 j=1,6
    if (id(j,i).ne.0) then
      write(out,*) i,j,id(j,i),mass(jdiag(id(j,i)))
    endif
450 continue
```

400 continue

return  
end

C=End Fortran

File: singeig.f

SUBROUTINE SSPACE (AK,AM,VL,VR,AKK,AMM,XX,EIGV,EIGOLD,ISL,  
1 IDIAG,NEIG,NVEC,NDOF,TOLEIG,TOLJAC,NW)

-----  
c  
c input :  
c  
c AK : stiffness matrix (profile values) (NDOF)  
c AM : consistent mass matrix (profile values) (NDOF)  
c ISL : stores in position "i" the row # of tip of  
c column "i" (NDOF)  
c IDIAG : position of diagonal terms in profile (NDOF)  
c NEIG : # of required eigenvalues  
c NVEC : # of subspace vectors  
c NDOF : # of degrees of freedom  
c TOLEIG : tolerance for eigenvalues convergence  
c TOLJAC : tolerance for Jacobi convergence  
c NW : logical unit number for output  
c  
c output:  
c  
c VL(NDOF,NVEC) : working array  
c - VL(.,1..NRMOD) : AM times rigid modes  
c - VL(.,NRMOD..NVEC) : subspace at the previous step  
c  
c VR(NDOF,NVEC) : eigen-vectors  
c - VR(.,1..NRMOD) : rigid modes  
c - VR(.,NRMOD..NVEC) : subspace at this step (eigenvectors)  
c  
c AKK(NVEC\*(NVEC + 1)/2) : stiffness matrix in the subspace  
c AMM(NVEC\*(NVEC + 1)/2) : consistent mass matrix in the subspace  
c XX(NVEC\*NVEC) : subspace eigenvectors  
c  
c EIGV(NVEC) : current eigenvalues  
c - EIGV(1..NRMOD) : 0 eigen-values  
c - EIGV(NRMOD..NVEC) : following eigenvalues > 0  
c  
c EIGOLD(NVEC) : same as EIGV  
c  
c-----

IMPLICIT REAL\*8 (A-H,O-Z)  
DIMENSION AK(1),AM(1),VL(NDOF,NVEC),VR(NDOF,NVEC),AKK(1),  
1 AMM(1),XX(NVEC\*NVEC),EIGV(1),EIGOLD(1),ISL(1),IDIAG(1)

C

WRITE (NW,1003) NEIG,NVEC,NDOF,TOLEIG

C

CALL INVECT (AK,AM,VL,VR,IDIAG,NDOF,NVEC)

CALL FACT (AK,IDIAG,ISL,NDOF,NW)

CALL NULL (AK,AM,VL,VR,IDIAG,ISL,NDOF,NRMOD)

```

C
C      NRMOD : # of rigid modes
C
C      WRITE (NW,1004) NRMOD
C      NSUB=NVEC-NRMOD
C
C      CALL ORTHO (VR,VL,NDOF,NRMOD,NVEC)
C
C      NIT=0
C      NSMAX=15
C      NITMAX=16
C      NVEC1=NVEC-1
C      DO 5 I=1,NVEC
5      EIGOLD(I)=0.0
C
C      500 NIT=NIT+1
C      WRITE (NW,1000) NIT
C
C      CALL SOLVES (AK,VL(1,NRMOD+1),VR(1,NRMOD+1),IDIAG,ISL,NDOF,NSUB)
C      CALL ORTHO (VL,VR,NDOF,NRMOD,NVEC)
C
C      IJ=0
C      DO 10 J=NRMOD+1,NVEC
C
C      CALCULATE THE UPPER PART OF AKK (SYMMETRIC)
C
C      DO 10 I=NRMOD+1,J
C      TR=0.0
C      DO 11 K=1,NDOF
11      TR=TR+VL(K,I)*VR(K,J)
C      IJ=IJ+1
C      AKK(IJ)=TR
C      10 CONTINUE
C
C      CALL MULT (AM,VR(1,NRMOD+1),VL(1,NRMOD+1),ISL,IDIAG,NSUB,NDOF)
C
C      IJ=0
C      DO 20 J=NRMOD+1,NVEC
C
C      CALCULATE THE UPPER PART OF AMM (SYMMETRIC)
C
C      DO 20 I=NRMOD+1,J
C      TR=0.0
C      DO 21 K=1,NDOF
21      TR=TR+VL(K,I)*VR(K,J)
C      IJ=IJ+1
C      AMM(IJ)=TR
C      20 CONTINUE
C
C      CALL JACOBI (AKK,AMM,XX,EIGV(NRMOD+1),NSMAX,TOLJAC,NSUB,NW)
C
C      ORDER EIGENVALUES & EIGENVECTORS
C
C      30 IS=0
C      DO 40 I=NRMOD+1,NVEC1
C      IF (EIGV(I+1).GE.EIGV(I)) GO TO 40
C      IS=1
C      TR=EIGV(I+1)
C      EIGV(I+1)=EIGV(I)

```



```

      EIGV(I)=TR
      DO 41 J=1,NSUB
        TR=XX(J+(I-NRMOD)*NSUB)
        XX(J+(I-NRMOD)*NSUB)=XX(J+(I-NRMOD-1)*NSUB)
41      XX(J+(I-NRMOD-1)*NSUB)=TR
40 CONTINUE
      IF (IS.EQ.1) GO TO 30

C
C SUBSPACE CONVERGENCE TEST
C
      ICONV=0
      DO 50 I=NRMOD+1,NVEC
        TR=DABS((EIGOLD(I)-EIGV(I))/EIGV(I))
        EIGOLD(I)=EIGV(I)
        EIGV(I)=TR
        IF (TR.GT.TOLEIG.AND.I.LE.NEIG) ICONV=1
50 CONTINUE
      WRITE (NW,1001) (EIGV(I),I=1,NVEC)
      IF (ICONV.EQ.0) GO TO 100

C
      IF (NIT.LE.NITMAX) GO TO 70
      WRITE (NW,1002)
      GO TO 100

C
C UPDATE EIGEN VECTORS
C
70 DO 80 I=1,NDOF
      DO 80 J=1,NSUB
        TR=0.0
        DO 81 K=1,NSUB
91      TR=TR+VR(I,K+NRMOD)*XX(K+(J-1)*NSUB)
          VL(I,J+NRMOD)=TR
80 CONTINUE
      DO 90 I=1,NDOF
        DO 90 J=NRMOD+1,NVEC
90      VR(I,J)=VL(I,J)
      GO TO 500

C
C CALCULATE FINAL EIGENVECTORS
C
100 DO 110 I=1,NDOF
      DO 110 J=1,NSUB
        TR=0.0
        DO 111 K=1,NSUB
111      TR=TR+VL(I,K+NRMOD)*XX(K+(J-1)*NSUB)
          VR(I,J+NRMOD)=TR
110 CONTINUE
      DO 112 I=1,NVEC
112      EIGV(I)=EIGOLD(I)

C
      RETURN

C
1000 FORMAT (5X,12HITERATION NO,I5)
1001 FORMAT (6(2X,1PE10.3))
1002 FORMAT (5X,24HWE ACCEPT CURRENT VALUES)
1003 FORMAT (//20X,'SUBSPACE ITERATION ROUTINE'///' NB OF EIGENVALUES=',
  1 I5/' NB OF VECTOR=',I5/' NB OF DOF=',I5/' TOLERANCE=',1PE10.3/)
1004 FORMAT (' NB OF RIGID NODES=',I5//)

```

```

END
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
SUBROUTINE INVECT (AK,AM,VL,VR,IDIAG,NDOF,NVEC)
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION AK(1),AM(1),VL(1),VR(NDOF,NVEC),IDIAG(1)
C
ND=NDOF/NVEC
C
DO 10 I=1,NDOF
  II=IDIAG(I)
  VR(I,1)=AM(II)
  VL(I)=AM(II)/AK(II)
  DO 10 J=2,NVEC
    VR(I,J)=0.0
10 CONTINUE
C
LL=NDOF-ND
C
DO 20 J=2,NVEC
  TR=0.0
C
  DO 30 I=1,LL
    IF (VL(I).LT.TR) GO TO 30
    TR=VL(I)
    IJ=I
30 CONTINUE
C
  DO 40 I=LL,NDOF
    IF (VL(I).LE.TR) GO TO 40
    TR=VL(I)
    IJ=I
40 CONTINUE
C
  VL(IJ)=0.0
  LL=LL-ND
  VR(IJ,J)=1.0
C
20 CONTINUE
C
RETURN
END
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
SUBROUTINE MULT (AM,VR,VL,ISL,IDIAG,NVEC,NDOF)
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION AM(1),VR(NDOF,NVEC),VL(NDOF,NVEC),ISL(1),IDIAG(1)
DO 500 IV=1,NVEC
  DO 100 I=1,NDOF
    TR=0.0
    IJ=IDIAG(I)
    IK=ABS(ISL(I))
    KK=I
    DO 110 K=IK,I
      TR=TR+AM(IJ)*VL(KK,IV)
      IJ=IJ-1
      KK=KK-1
110 CONTINUE
C
IF (I.EQ.NDOF) GO TO 99

```



```

D1=AB+SQCH
D2=AB-SQCH
DEN=D1
IF (DABS(D2).GT.DABS(D1)) DEN=D2
IF (DEN.NE.ZERO) GO TO 45
CA=ZERO
CG=-AK(JK)/AK(KK)
GO TO 50
45 CA=AKK/DEN
CG=-AJJ/DEN

```

C  
C  
C

PERFORM GENERALIZED ROTATION

50

```

JP1=J+1
JM1=J-1
KP1=K+1
KM1=K-1
IF (JM1.LT.1) GO TO 70
DO 60 I=1,JM1
IJ=J+JM1/2+1
IK=K+KM1/2+1
AKJ=AK(IJ)
AKK=AK(IK)
AMJ=AM(IJ)
AMK=AM(IK)
AK(IJ)=AKJ+CG+AKK
AM(IJ)=AMJ+CG+AMK
AK(IK)=AKK+CA+AKJ
AM(IK)=AMK+CA+AMJ

```

60

CONTINUE

70

```

IF (KP1.GT.N) GO TO 90
DO 80 I=KP1,N
JI=I+(I-1)/2+J
KI=I+(I-1)/2+K
AKJ=AK(JI)
AMJ=AM(JI)
AKK=AK(KI)
AMK=AM(KI)
AK(JI)=AKJ+CG+AKK
AM(JI)=AMJ+CG+AMK
AK(KI)=AKK+CA+AKJ
AM(KI)=AMK+CA+AMJ

```

80

CONTINUE

90

```

IF (JP1.GT.KM1) GO TO 110
DO 100 I=JP1,KM1
JI=I+(I-1)/2+J
IK=K+(K-1)/2+I
AKJ=AK(JI)
AMJ=AM(JI)
AKK=AK(IK)
AMK=AM(IK)
AK(JI)=AKJ+CG+AKK
AM(JI)=AMJ+CG+AMK
AK(IK)=AKK+CA+AKJ
AM(IK)=AMK+CA+AMJ

```

100

CONTINUE

110

```

AKK=AK(KK)
AMK=AM(KK)
AKJ=AK(JJ)

```

```

      AMJ=AM(JJ)
      AK(KK)=AKK+TWO*CA*AK(JK)+CA*CA*AKJ
      AM(KK)=AMK+TWO*CA*AM(JK)+CA*CA*AMJ
      AK(JJ)=AKJ+TWO*CG*AK(JK)+CG*CG*AKK
      AM(JJ)=AMJ+TWO*CG*AM(JK)+CG*CG*AMK
      AK(JK)=ZERO
      AM(JK)=ZERO
C
C      UPDATE EIGENVECTOR FOR THIS ROTATION
C
      DO 120 I=1,N
          XXJ=XX(I,J)
          XXK=XX(I,K)
          XI(I,J)=XXJ+CG*XXK
          XI(I,K)=XXK+CA*XXJ
120     CONTINUE
150     CONTINUE
C
C      UPDATE EIGENVALUES & CHECK CONVERGENCE
C
      NT=N*(N+1)/2
      ICONV=0
      DO 160 I=1,N
          II=I*(I-1)/2+I
          IF (AK(II).LE.ZERO.OR.AM(II).LE.ZERO) GO TO 900
          TR=AK(II)/AM(II)
          DEN=(TR-EIGV(I))/TR
          EIGV(I)=TR
          IF (DABS(DEN).GT.TOL) ICONV=1
160     CONTINUE
          IF (ICONV.EQ.1) GO TO 499
C
C      CHECK OFF DIAGONAL TERMS
C
      EPS=TOL*TOL
      DO 170 J=1,NR
          IIK=J+1
          DO 170 K=IIK,N
              JJ=J*(J-1)/2+J
              KK=K*(K-1)/2+K
              JK=K*(K-1)/2+J
              EPSAK=(AK(JK)*AK(JK))/(AK(JJ)*AK(KK))
              EPSAM=(AM(JK)*AM(JK))/(AM(JJ)*AM(KK))
              IF (EPSAK.LT.EPS.AND.EPSAM.LT.EPS) GO TO 170
          GO TO 499
170     CONTINUE
C
C      SCALE EIGENVECTORS
C
179     DO 180 I=1,N
          II=I*(I-1)/2+I
          AKK=DSQRT(AM(II))
          DO 180 J=1,N
              XX(J,I)=XX(J,I)/AKK
180     CONTINUE
C
      RETURN
C
499     IF (NSWEEP.LE.NSMAX) GO TO 500

```

```

WRITE (NW,1000)
GO TO 179
C
900 WRITE (NW,1001)
STOP
C
1000 FORMAT (5X,34HNO CONVERGENCE AT NMAX ITERATIONS)
1001 FORMAT (5X,46HERROR IN JACOBI : MATRIX NOT POSITIVE DEFINITE)
C
END
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
SUBROUTINE SOLVES (AK,VL,VR,IDIAG,ISL,NDOF,NVEC)
C
C PURPOSE : SOLVES THE SINGULAR PROBLEM : AK x VL = VR
C THE SINGULAR COLUMNS INTO AK ARE INDEXED BY
C THE NEGATIVE VALUES OF ISL, THE CORRESPONDING
C TERMS OF THE SOLUTION ARE PUT TO 0
C
C
C IMPLICIT REAL*8 (A-H,O-Z)
C DIMENSION AK(1),VL(NDOF,NVEC),VR(NDOF,NVEC),IDIAG(1),ISL(1)
C
DO 500 IV=1,NVEC
DO 50 I=1,NDOF
50 VL(I,IV)=VR(I,IV)
C
C BACKSUBSTITUTE
C
DO 100 IC=2,NDOF
TR=0.0
IC1=IC-1
IN1=IDIAG(IC)-IC
IK=ISL(IC)
IF (IK.LE.0) THEN
VL(IC,IV)=0.
GOTO 100
ENDIF
IF (IK.GT.IC1) GO TO 100
DO 120 K=IK,IC1
TR=TR+AK(IN1+K)*VL(K,IV)
120 CONTINUE
VL(IC,IV)=VL(IC,IV)-TR
100 CONTINUE
C
C SOLVE DU=U
C
DO 150 IC=1,NDOF
VL(IC,IV)=VL(IC,IV)/AK(IDIAG(IC))
150 CONTINUE
C
C BAKSUBSTITUTE
C
IIC=NDOF
DO 200 IC=2,NDOF
TR=VL(IIC,IV)
IC1=IIC-1
IK=ISL(IIC)
IN1=IDIAG(IIC)-IIC

```

```

        IF ((IK.GT.IC1).OR.(IK.LE.0)) GO TO 221
        DO 220 K=IK,IC1
            VL(K,IV)=VL(K,IV)-AK(IN1+K)*TR
220     CONTINUE
221     IIC=IIC-1
200     CONTINUE
500     CONTINUE
        RETURN
        END
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
SUBROUTINE FACT (AK,IDIAG,ISL,NDOF,NW)
C
C   PURPOSE : LDLT DECOMPOSITION OF THE POSITIVE SEMI-DEFINITE
C             MATRIX AK, THE SINGULAR COLUMNS OF AK ARE INDEXED
C             BY A NEGATIVE VALUE OF IDIAG>
C
C
IMPLICIT REAL*8 (A-H,O-Z)
DIMENSION AK(1),IDIAG(1),ISL(1)
C
C   DETERMINE MIN & MAX
C
TR=DABS(AK(1))
AMIN=TR
AMAX=TR
DO 10 IL=1,NDOF
    TR=DABS(AK(IDIAG(IL)))
    IF (TR.LT.AMIN) AMIN=TR
    IF (TR.GT.AMAX) AMAX=TR
10 CONTINUE
ZERO=(AMAX+AMIN)*1.0D-10
C
C   LOOP OVER COLUMN
C
DO 100 IC=1,NDOF
    MIC=ISL(IC)
    IC1=IC-1
    MIC1=MIC+1
    IN2=IDIAG(IC)-IC
    IF ((MIC.LT.1).OR.(MIC.GT.IC)) GO TO 901
C
C   CALCULATE GS
C
IF (MIC1.GT.IC1) GO TO 150
DO 120 IL=MIC1,IC1
    IF (IDIAG(IL).LT.0) THEN
        AK(IN2+IL)=0
        GOTO 120
    ENDIF
    MIL=ISL(IL)
    IL1=IL-1
    MIN=MAX0(MIL,MIC)
    IN1=IDIAG(IL)-IL
    TR=0.0
    IF (MIN.GT.IL1) GO TO 120
    DO 130 K=MIN,IL1
        TR=TR+AK(IN1+K)*AK(IN2+K)
130 CONTINUE

```

```

      IN=IN2+IL
      AK(IN)=AK(IN)-TR
120  CONTINUE
C
C   CALCULATE L&D
e
150  TR=0.0
      IF (NIC.GT.IC1) GO TO 201
      DO 200 IL=NIC,IC1
          IF (IDIAG(IL).LT.0) GOTO 200
          AG=AK(IN2+IL)
          AL=AG/AK(IDIAG(IL))
          AK(IN2+IL)=AL
          TR=TR+AL*AG
200  CONTINUE
201  IN=IDIAG(IC)
      AK(IN)=AK(IN)-TR
      IF (AK(IN).LT.ZERO) IDIAG(IC)=-IDIAG(IC)
100  CONTINUE
      RETURN

901  WRITE (NW,1001)
      STOP

1001 FORMAT (5X,29H***STOP ERROR IN IDIAG VECTOR)
1010 FORMAT (5X,'CONDITIONING OF THE STIFFNESS MATRIX'/2X,
1 'MIN DIAG TERM=',1PE10.3,' MAX DIAG TERM=',E10.3)

      END
CCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCCC
      SUBROUTINE NULL (AK,AM,VL,VR,IDIAG,ISL,NDOF,NRMOD)
C
C   PURPOSE : CALCULATE THE NULL SPACE OF AK AND PUT AN
C             ORTHONORMALISED BASE OF THIS SPACE INTO THE
C             NRMOD FIRST VECTORS OF VR.
C             VL = AM x VR AFTER EXECUTION
C
C
C   IMPLICIT REAL*8 (A-H,O-Z)
C   DIMENSION AK(1),AM(1),IDIAG(1),ISL(1),VL(NDOF,*),VR(NDOF,*)
C
C   STORE THE SINGULAR COLUMNS INTO THE BEGINNING OF VR
C
C   THE SINGULAR EQUATION ARE NOW INDEXED BY NEGATIVE VALUES
C   INTO ISL INSTEAD OF IDIAG
C
      NRMOD=0
      DO 1 IC=1,NDOF
          IF (IDIAG(IC).GT.0) GOTO 1
          IDIAG(IC)=-IDIAG(IC)
          NRMOD=NRMOD+1
          NIC=ISL(IC)
          IN=IDIAG(IC)-IC
          DO 2 K=1,NIC-1
              VR(K,NRMOD)=0.
          DO 3 K=NIC,IC-1
              VR(K,NRMOD)=AK(IN+K)
              AK(IN+K)=0.
          CONTINUE
      1
      2
      3

```



```

ISL(IC)=-ISL(IC)
VR(IC,NRMOD)=-1.
AK(IDIAG(IC))=1.
DO 4 K=IC+1,NDOF
  VR(K,NRMOD)=0.
4
1 CONTINUE

```

C  
C  
C

BAKSUBSTITUTE

```

DO 200 N=1,NRMOD
  IIC=NDOF
  DO 200 IC=2,NDOF
    TR=VR(IIC,N)
    IC1=IIC-1
    IK=ISL(IIC)
    IN1=IDIAG(IIC)-IIC
    IF ((IK.GT.IC1).OR.(IK.LE.0)) GO TO 221
    DO 220 K=IK,IC1
      VR(K,N)=VR(K,N)-AK(IN1+K)*TR

```

```

220 CONTINUE
221 IIC=IIC-1
200 CONTINUE

```

C  
C  
C

ORTHOGONALISATION

```

DO 10 N=1,NRMOD
  DO 20 K=1,N-1
    TR=0.
    DO 30 I=1,NDOF
      TR=TR+VL(I,K)*VR(I,N)
    DO 40 I=1,NDOF
      VR(I,N)=VR(I,N)-TR*VR(I,K)
    CONTINUE
    CALL MULT(AM,VL(1,N),VR(1,N),ISL,IDIAG,1,NDOF)
    TR=0.
    DO 50 I=1,NDOF
      TR=TR+VR(I,N)*VL(I,N)
    TR=1/SQRT(TR)
    DO 60 I=1,NDOF
      VR(I,N)=VR(I,N)*TR
      VL(I,N)=VL(I,N)*TR
    CONTINUE
  CONTINUE
10 CONTINUE

```

RETURN  
END

CC

SUBROUTINE ORTHO(VL,VR,NDOF,NRMOD,NVEC)

C  
C  
C  
C  
C  
C

PURPOSE : ORTHOGONALISE THE NSUB LAST COLUMNS OF VL (LAST  
EVALUATED SOLUTION) WITH RESPECT TO THE NULL SPACE  
OF A, FOR THE AM SCALAR PRODUCT

IMPLICIT REAL\*8 (A-H,O-Z)  
INTEGER NDOF,NRMOD,NVEC  
DIMENSION VL(NDOF,NVEC),VR(NDOF,NVEC)  
  
NSUB=NVEC-NRMOD

```

DO 1 J=1,NSUB
  DO 2 I=1,NRMOD
    S=0.
    DO 3 K=1,NDOF
      3   S=S+VL(K,I)*VL(K,NRMOD+J)
        DO 4 L=1,NDOF
          4   VL(L,NRMOD+J)=VL(L,NRMOD+J)-S*VR(L,I)
        2   CONTINUE
      1   CONTINUE
    RETURN
  END

```

CC

File: animout.f

C=Module ANIMOUT  
C=Author K. Alvin  
C=Date June 1990  
C=Block Fortran

```

C -----
C
C   subroutine ANIMOUT
C
C   Purpose:
C     This subroutine produces an output file to be used to
C     visualize the simulation using MESH.
C -----
C

```

```

subroutine ANIMOUT(q,id,nnp,time,out)
real*8 q(1),time,v(3)
integer out,id(6,'),nnp,i,k
write(out,'(g15.5)') time
do 100 i=1,nnp
  do 200 k=1,3
    if (id(k,i) .ne. 0) then
      v(k) = q(id(k,i))
    else
      v(k) = 0.
    endif
  200 continue
  write(out,1000) (v(k),k=1,3)
100 continue
1000 format(3g15.5)
return
end

```

File: stiffrc.f

```

C   subroutine STIFFRC(v,fact,kq)

```

```

recursive subroutine STIFFRC(v,fact,kq)

real*8 v(1),fact,kq(1)

include 'shared.inc'

C ASSEMBLE EACH ELEMENT MASS AND STIFFNESS

do 100 ii=1,ndomain

CVD$ CMCALL
do 110 jj=1,neld(ii)
n = elnum(jj,ii)
call ELEFRC(v,fact,kq,n)
110 continue

100 continue

return
end

C subroutine ELEFRC(v,fact,kq,n)

recursive subroutine ELEFRC(v,fact,kq,n)

real*8 v(1),fact,kq(1)
integer n

include 'shared.inc'

C LOCAL VARIABLES

parameter(MAXSEQ=24)
real*8 sk(MAXSEQ,MAXSEQ)
integer lm(MAXSEQ),nseq

do 20 k=1,4
j=ix(k,n)
if ((etype(n).eq.1).and.(k.gt.2)) j = 0
do 30 i=1,6
kk=6*(k-1) + i
if (j .ne. 0) then
lm(kk) = id(i,j)
else
lm(kk) = 0
endif
30 continue
20 continue

nseq=12

call LOADSK(sk,n,nseq)

call ESTIFVH(sk,lm,nseq,v,kq,fact)

return
end

```

```

C      subroutine LOADSK(sk,n,nseq)

      recursive subroutine LOADSK(sk,n,nseq)

      include 'shared.inc'

      real*8 sk(nseq,1)
      integer n,nseq

      k=0
      do 10 j=1,nseq
        do 20 i=1,j
          k=k+1
          sk(i,j)=estifm(k,n)
          sk(j,i)=sk(i,j)
20      continue
10     continue

      return
      end

```

---

File: estifvm.f

---

```

C      subroutine ESTIFVM(sk,lm,nseq,v,kq,fact)

      recursive subroutine ESTIFVM(sk,lm,nseq,v,kq,fact)

C     ARGUMENTS

      real*8 sk(nseq,1),v(1),kq(1),fact
      integer lm(1),nseq

      do 20 j = 1, nseq
        k = lm(j)
        if (k .eq. 0) goto 20
        do 10 i = 1, nseq
          m = lm(i)
          if (m .eq. 0) goto 10
          kq(m) = kq(m) + sk(i,j)*v(k)*fact
10      continue
20     continue

      return
      end
C=End Fortran

```

---

File: renum.f

---

```

C=DECK RENUM
C=PURPOSE- RENUMBERS THE GRID POINTS TO MINIMIZE PROFILE STORAGE
C=AUTHOR W K BELVIN and DUC NGUYEN 7-5-90
C
      subroutine RENUM
C
      include 'shared.inc'

```

```

C
C   Initialize vector
C
   maxtry=2*nnp/3 +1
c.....
   nterms=nnp*maxtry
   do 22 j=1,nterms
22  iadjcy(j)=0
   do 10 i=1,nnp
10  icount(i)=0
c*****
   do 1 i=1,nel
     nodea=ix(1,i)
     nodeb=ix(2,i)
     if ((nodea .eq. 0).or.(nodeb .eq. 0)) go to 1
     icount(nodea)=icount(nodea)+1
     icount(nodeb)=icount(nodeb)+1
     ia=icount(nodea)
     ib=icount(nodeb)
     if(ia.gt.maxtry .or. ib.gt.maxtry) go to 345
     locate=(nodea-1)*maxtry+ia
     iadjcy(locate)=nodeb
     locate=(nodeb-1)*maxtry+ib
     iadjcy(locate)=nodea
1   continue
c*****
   ii=0
   do 37 i=1,nterms
     if ( iadjcy(i) .eq. 0 ) go to 37
     ii=ii+1
     iadjcy(ii)=iadjcy(i)
37  continue
c*****
   last=0
   do 2 i=1,nnp
     last=last+icount(i)
2   continue
   jj=icount(1)
   icount(1)=1
   do 3 i=2,nnp+1
     kk=icount(i)
     icount(i)=icount(i-1)+jj
     jj=kk
3   continue
   go to 556
345 write(6,555)
555 format(2x,'error in dimension for MAXTRY !! ')
556 continue
c*****
   call GENRCM(nnp,icount,iadjcy,perm,mask,xls)
   return
   end
cXXXXXXXXXXXXXXXXXXXXX
   subroutine genrcm(neqns,radj,adjncy,perm,mask,xls)
c.....reference: computer solution of large sparse positive definite
c..... systems, alan george & joseph w-h liu
c..... (prentive-hall,inc.,englewood cliffs,NJ 07632)
   integer adjncy(1),mask(1),perm(1),xls(1)

```

```

integer xadj(1),ccsize,i,neqns,nlvl,num,root
do 100 i=1,neqns
  mask(i)=1
100  continue
  num=1
  do 200 i=1,neqns
    if( mask(i).eq.0 ) go to 200
    root=i
    call fnroot(root,xadj,adjncy,mask,nlvl,xls,perm(num) )
    call rcm(root,xadj,adjncy,mask,perm(num),ccsize,xls)
    num=num+ccsize
    if(num.gt.neqns) go to 987
200  continue
c.....perm(new node)=old node
c.....now, mask(old node)= new node
987  continue
  do 11 new=1,neqns
    iold=perm(new)
    mask(iold)=new
c    write(6,*) 'iold,mask(iold) = ',iold,mask(iold)
11  continue
  return
end
c*****
subroutine fnroot(root,xadj,adjncy,mask,nlvl,xls,ls)
integer adjncy(1),ls(1),mask(1),xls(1)
integer xadj(1),ccsize,j,jstrt,k,kstop,kstrt,
$   mindeg,nabor,ndeg,nlvl,node,nunlvl,root
call rootls(root,xadj,adjncy,mask,nlvl,xls,ls)
ccsize=xls(nlvl+1)-1
if(nlvl.eq.1 .or. nlvl.eq.ccsz) return
100  jstrt=xls(nlvl)
  mindeg=ccsize
  root=ls(jstrt)
  if(ccsize.eq.jstrt) go to 400
  do 300 j=jstrt,ccsize
    node=ls(j)
    ndeg=0
    kstrt=xadj(node)
    kstop=xadj(node+1)-1
    do 200 k=kstrt,kstop
      nabor=adjncy(k)
      if( mask(nabor) .gt. 0) ndeg=ndeg+1
200  continue
    if(ndeg.ge.mindeg) go to 300
    root=node
    mindeg=ndeg
300  continue
400  call rootls(root,xadj,adjncy,mask,nunlvl,xls,ls)
  if(nunlvl.le.nlvl) return
  nlvl=nunlvl
  if(nlvl.lt.ccsz) go to 100
  return
end
c*****
subroutine rcm(root,xadj,adjncy,mask,perm,ccsize,deg)
integer adjncy(1),deg(1),mask(1),perm(1)
integer xadj(1),ccsize,inbr,i,j,jstop,jstrt,k,l,lbegin,
$   lnbr,lperm,lvlend,nbr,node,root

```

```

call degree(root,xadj,adjncy,mask,deg,ccsize,perm)
mask(root)=0
if(ccsize.le.1) return
lvlend=0
lnbr=1
100 lbegin=lvlend+1
    lvlend=lnbr
    do 600 i=lbegin,lvlend
        node=perm(i)
        jstrt=xadj(node)
        jstop=xadj(node+1)-1
        fnbr=lnbr+1
        do 200 j=jstrt,jstop
            nbr=adjncy(j)
            if(mask(nbr).eq.0) go to 200
            lnbr=lnbr+1
            mask(nbr)=0
            perm(lnbr)=nbr
200        continue
        if(fnbr.ge.lnbr) go to 600
        k=fnbr
300        l=k
            k=k+1
            nbr=perm(k)
400        if(l.lt.fnbr) go to 500
            lperm=perm(l)
            if( deg(lperm).le.deg(nbr) ) go to 500
            perm(l+1)=lperm
            l=l-1
            go to 400
500        perm(l+1)=nbr
            if(k.lt.lnbr) go to 300
600        continue
        if(lnbr.gt.lvlend) go to 100
        k=ccsize/2
        l=ccsize
        do 700 i=1,k
            lperm=perm(l)
            perm(l)=perm(i)
            perm(i)=lperm
            l=l-1
700        continue
        return
    end
cXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXXX
subroutine rootls(root,xadj,adjncy,mask,nlvl,xls,ls)
integer adjncy(1),ls(1),mask(1),xls(1)
integer xadj(1),i,j,jstop,jstrt,lbegin,ccsize,lvlend,
$      lvsize,nbr,nlvl,node,root
mask(root)=0
ls(1)=root
nlvl=0
lvlend=0
ccsize=1
200 lbegin=lvlend+1
    lvlend=ccsize
    nlvl=nlvl+1
    xls(nlvl)=lbegin
    do 400 i=lbegin,lvlend

```

```

node=ls(i)
jstrt=xadj(node)
jstop=xadj(node+1)-1
if(jstop.lt.jstrt) go to 400
do 300 j=jstrt,jstop
  nbr=adjncy(j)
  if( mask(nbr).eq.0) go to 300
  ccsize=ccsize+1
  ls(ccsize)=nbr
  mask(nbr)=0
300  continue
400  continue
  lvsize=ccsize-lvlend
  if(lvsize.gt.0) go to 200
  xls(nlvl+1)=lvlend+1
  do 500 i=1,ccsize
    node=ls(i)
    mask(node)=1
500  continue
  return
end
CXXXXXXXXXXXXXXXXXXXXXXXXXXXX
subroutine degree(root,xadj,adjncy,mask,deg,ccsize,ls)
integer adjncy(1),deg(1),ls(1),mask(1)
integer xadj(1),ccsize,i,ideg,j,jstop,jstrt,
$   lbegin,lvlend,lvsize,nbr,node,root
ls(1)=root
xadj(root)=-xadj(root)
lvlend=0
ccsize=1
100  lbegin=lvlend+1
  lvlend=ccsize
  do 400 i=lbegin,lvlend
    node=ls(i)
    jstrt=-xadj(node)
    jstop=iabs( xadj(node+1) ) -1
    ideg=0
    if(jstop.lt.jstrt) go to 300
    do 200 j=jstrt,jstop
      nbr=adjncy(j)
      if( mask(nbr).eq.0 ) go to 200
      ideg=ideg+1
      if(xadj(nbr).lt.0) go to 200
      xadj(nbr)=-xadj(nbr)
      ccsize=ccsize+1
      ls(ccsize)=nbr
200  continue
300  deg(node)=ideg
400  continue
  lvsize=ccsize-lvlend
  if(lvsize.gt.0) go to 100
  do 500 i=1,ccsize
    node=ls(i)
    xadj(node)=-xadj(node)
500  continue
  return
end

```

File: kfilter.f



```

C      subroutine KFILTER

      recursive subroutine KFILTER

      include 'shared.inc'

      do 10 i = 1,ndof
        go(i) = delsq*(f(i)+go(i))+delta*pe(i)+mass(jdiag(i))*qe(i)
        gk(i) = delta*(f(i)+gk(i))+pe(i)
10     continue

      call SOLVER(eo,go,jdiag,ndof,2)

C      Activate EBE computations for internal force by using STIFFRC
C      subroutine.  Otherwise use PMVHAD (profile matrix/vector mult-add)

C      call PMVHAD(stif,jdiag,go,ndof,-delta,gk,1.d0)
      call STIFFRC(go,-delta,gk)

      do 100 i = 1,ndof
        qe(i) = 2.*go(i) - qe(i)
        pe(i) = 2.*gk(i) - pe(i)
100    continue

      return
      end

```