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THE IATERAI FAIIURE OF SPARS
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## NATIONAL ADVISORY COMITTTEE FOR AERONAUTICS.

TECHIICAI NOTE NO. 232.

## THE LATERAT FAILURE OF SPARS.

By Stevens Eromley and William H. Robinson, Jr.

In the design of spars it often happens that the ajrfoil section will permit the use of a decper spar than necessary for sufficient strength if the depth-oreadth ratio is to be kept within the limits conventional in beams, limits generally obscrved as permitting the uso in computation of the ordinary beam formula:

$$
f=\frac{\mathbb{M} Y}{I}
$$

In the case of rectangular sections

$$
f=\frac{M \frac{d}{2}}{\frac{b d^{3}}{12}}=\frac{6 M}{b d^{2}}
$$

where

$$
\begin{aligned}
& b=\text { spar breadth } \\
& d=\text { spar depth }
\end{aligned}
$$

From the above it is observed that the strength of a spar varies as the square of its depth. Since the weight of a spar varies as the first power of the dopth, tho maximum strenothweight ratio will be gainod by the uso of as decp a spar as possiblc, other factors being equal.

* Work done as a thesis in aeronautical ongineering at the Massachusetts Institute of Technology.

When a spar of very large depth-breadth ratio is subjected to bending, for a time it acts as a beam according to beam formulae. At some point as the load is increased, however, that part of the spar under compression from the bending begins to buckle as a column and deflects laterally. More lateral deflection accompanies any further increase in the load until a maximum load is reached and the beam fails, a load considerably below that directly computed from the beam formula.

The fibers of the spar under maximum tension from the bending remain straight. As viewed from the end there is no apparent distortion of the sectional form at any point, but a simpile torsion.

If the stress-strain diagram be plotted, it is found to be of the general shape shown in Fig. I.

Fig. 1


From $O$ to $A$ the spar acts like an elastic material. At A, where the calculated stress is still well below the elastic
limit, lateral deflection sets in and continues until the spocimen fails either under compression from the primary bending or a combination of primary and lateral bending, or under tension or, most probably, until the excossive deflection causes secondary structural members to fail and the structure to disintegrate.

But little rescarch has been made on this subject so far as could be ascertaincd. There are, however, a few sets of tests which covered some parts of the present work.

As for the mathematical analysis of this subject, there has been one treatment specifically directed to the attention of the deronautical engineer. In "Flight," May 30, 1918, there appeared a note by J. Prescott, M.A., D.Sc., entitled "The Sideways Buckling of Loaded Beams of Deep Section." The details of his work are not available, but the method implied suggested a mechanical analysis of the question rather than any experimental work. To quote Prescott, "The buckling load depends on the flexural rigidity for sideways bending, and on the torsional rigidity of the bcam." The latter is true in that a beam could not buckle without twisting (Reference l). Prescott published a very interesting formula by which the ultimate load can be computed, and which takes, for a simply loaded beam, the form

$$
P=\frac{16.94 \sqrt{E I \mathbb{N}}}{I^{2}}
$$

where

$$
\begin{aligned}
& P=\text { concentrated load at the center. } \\
& L=\text { length of the beam. } \\
& \mathbb{E}=\text { modulus of elasticity. } \\
& I=\text { smallest moment of inertia of the section. } \\
& \mathbb{N}=\text { modulus of rigidity. } \\
& \mathrm{KN}=\text { torsional rigidity. }
\end{aligned}
$$

From the theory of the torsion of prisms, approximately:

$$
\left.K=\frac{3 b^{3} y^{3}}{10\left(b^{2}+h^{2}\right)} \text { (Reference } 2\right)
$$

$$
b=\text { breadth of beam. }
$$

$$
h=\text { depth of beam. }
$$

In all cases Prescott considerod the load applied at the center line of the beam.

## Nature of Experiments

Prescott's theory prescribes conditions of loading, and in order to check this formula experimentally, special precautions were taken to insure these conditions. For the determination of the relation between depth-breadth ratio and lateral failure the specimens were supported at the ends so that they were free to deflect in their own plane, and partially free to deflect laterally. The ends were mounted on rollers so that there could be no external horizontal forces apolied to the bcara.

In studying the offect of span only threc specimens wore used, each being tested at several different lengths. The apparatus was as described above, and the span ( $l$ ) was shortened by moving both end yokes toward the center. This was possible because failures by lateral collapse occurred at stresses below the elastic limit of the material, and repeated tosts could therefore bo made on a singlo specimen.

For the determination of the effect of load applied at more than one point, undamaged specimens were tested with loads at the third points.

The wood used in these tests was western spruce, kiln dried, but only of fair quality. Although all specimens came from the same source and apparently from the same tree, some of the grain slopes werc excessive.

The sizes selected were such as to fit the apparatus. Three specimens of each size were used. Proportions were varied through a range wido cnough to insure that both lateral and direct failures would occur. In general, the dimensions of tho sections within any group were varied so that the section modulus would remain substantially constant.

All specimens were 48 inches in length, except the ones used for the span tests, which were 58 inches long. All told, 54 tests were made on 27 specimens.

The load was transmitted to the specimen through a yoke and distributed by steel and wooden blocks over a portion of
the length of the beam sufficient to prevent local crushing. The effect of this distribution of load, reducing the maximum bending moment by approximately one percent, is negligible when compared to factors such as the variation in the wood, etc. The yokes fitted closely along the sides of the specimens to prevent lateral deflection at any point of load application, being further filled in with paper shims. Some of the load was therefore transmitted through the sides of the specimen. The supporting yokes were similar, but usually no distribution of the load was necessary at those points.

When lateral deflection set in it continued until the boam of the testing-machine dropped. Beyond this point no more load could be applicd, the beam simply distorting further and further. The point at which the beam dropped therefore gave the maximum load.

If the specimen failed in tension the failure load was recorded. If it showed evidence of crushing, more load was applied until the beam failed in tension or a maximum load was reached.

In testing some of the heavier specimens crushing appcared at the end supports. Wooden blocks $1 / 2^{\prime \prime} \times 3^{\prime \prime} \times 1 / 16^{\prime \prime}$ were used for wider distribution of the load with these supports, and when heavier loads yet were applied steel ones $1^{\prime \prime} \times 5^{\prime \prime} \times$ 1/2" were introduced. The wooden blocks are tabulated as 1-1, stecl 3-3.

## Corroction of Data

In the correction of the data for the modulus of rupture, the following assumptions have boen made:
(1) That specific gravity is a function of percent summer growth and rate of growth, and that a correction for specific gravity will include the two latter.
(2) That moisture content, grain slope, and specific gravity, while affecting the modulus of rupture, do not alter the tendency to fail laterally. This assumption means that lateral failure is governed only by the dimensions of the specimen and the manner of loading:
(3) That moisture, grain slope, and specific gravity affect the modulus of rupture the same, whether the specimen fails laterally or not.

The above assumptions apply also to the modulus of elasticity corrections. The methods of correction used were drawn from Bulletin No. 70 and Project Report No. 2284 of the Forest Products Laboratory. Views of the Laboratory and testingmachine are shown in Figs. 2 and 3. From this corrected moduIus of rupture a corrected maximum bending moment ( $M_{C}$ ) was obtained. This was further corrected $\left(M_{c}{ }^{1}\right)$ to a standard. sectional area by multiplying by the three halves power of the ratio between the sectional area of the specimen and a standard value.

## Standard Values

The average moisture content of the specimens was $7.36 \%$, the average specific gravity .396. All results were corrected to these values and to a zero grain slope.

Discussion of Results

In the plots which were constructed all values to the left of the dotted vertical line which has been drawn represent tension or compression failures, while those to the right represent lateral failures. There were no overlaps, the division betwoon the two types of failure in terms of depth-breadth ratio being sharply defined.

Fig. 4 is a plot of the depth-breadth ratio against corrected modulus of rupture for beams of constant span with a concentrated load at the center. The low points at depthbreadth ratio of about 4 and 10 represent single specimens, presumably of poorer than average material.

The curve best representing the points on this plot is, it will be observed, one nearly horizontal to the left of the dotted line and dropping sharply down to the right in the lateral failure region. Fig. 5 is a like plot for the span tests and is quite similar.

Fig. 6 is a plot of the depth-breadth ratio vs. the corrected modulus of rupture for the tests with third-point load-
ing. The curve is of like trend as the two preceding. The spocimon indicated by the cross and arrow failed in tension instead of laterally. The failure occurred in a region of sapwood, presumably before lateral deflection had started. The mean curve in Fig. 6, so far as it extends, is almost identical with that in Fig. 4.

In like manner Fig. 7 has been plotted for corrected bending moments reduced to a constant sectional area of 2.46 sq .in., the average for the specimens. It will be observed from Fig. 7 that the moment reaches a maximum at a depth-breadth ratio of about 7. It would therefore be inadvisable to permit the ratio to exceed this figure in a beam, however great the depth that might be available. In a wing spar, however, a still larger ratio would be permissible because of the added lateral support given by the ribs. There is no explanation excepting a defective specimen, as to why the bending moment should again fall off at depth-breadth ratio of 4 .

Fig. 9 was plotted to show the modulus of elasticity variation with depth-breadth ratio for the single-load tests.

Fig. 10 is a plot of depth-breadth ratio Vs. span-breadth ratio ( $l / b$ ) for all the specimens loaded at the middle point. The number adjacent to each point represents in approximate thousands of pounds per square-inch, the modulus of rupture of that specimen. The dotted line is drawn through the point which represents specimen 9 B of 30 -inch span which failod in
compression and laterally at the same time. A negatively sloping line such as the dotted one shown, dividos the causes of failure prociscly, those above and to the right boing latcral failures and those below and to the loft boing either tonsion or comprossion failuros.

An attompt was mado to chock Prescott's formula for a beam simply loaded and failing laterally:

$$
P=\frac{16.94}{I^{2}} \sqrt{E I N K}
$$

Three representative tests were chosen, and in all cases the specimen failed at a much lower load than that computed from Prescott's formula, ranging from one-half to one-fifth the computed value ( $N$ being taken as 90,000 ). This is somewhat surprising, as tests on steel beams have shown excellent agrecment with the figures given by the formula. Tho discrepancy so may be due to the homogencous and isotropic nature of the metal and the quite different structure of the wood. Such other lateral failure tests as have previously been made on wood seem to agree with this work in making the importance of lateral failure appear greater in practice than the theory would indicate.

From Fig. 4 it is seen that the strength of tho specimen as denoted by its modulus of rupture increases as the depthbreadth ratio decreases. From Fig. 5 the modulus of rupture increases as the span-depth ratio decreases.

From Fig. 10 it is observed that the tendency to fail laterally does not bear a constant relation to the modulus of rupture.

The conclusion from those tosts is that after the critical span or depth-breadth ratio has boen roached, the modulus of rupturc varios approximately inversely as the first power of the span and of the depth-breadth ratio.

The direction of lateral deflection is alternate botwoon succossive supports by theory and all tests. For this reason we believe that rib spacing along the spar is more important in reducing latoral doflcction than the distance botween supports at the strut points. Furthermore, we belicve that within the limits of modern dosign any increase in distance botwoen strut points can well be compensated for by spacing the ribs closcr togethor, providing tho ribs do furnish lateral support.

## Tablo I.

Charactoristics of the Specimens

| $\begin{aligned} & \text { Spec- } \\ & \text { imen } \end{aligned}$ | b | h | $\mathrm{h} / \mathrm{b}$ | y/I | I S | Slope | \%SG | \% 1 | RG | SG |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $1 \mathrm{~A}^{\prime}$ | . 53 | 6.00 | 11.32 | . 315 | 9.54 | 50 | 40 | 10.20 | 18 | . 397 |
| $10^{\prime}$ | . 50 | 5.88 | 11.76 | . 347 | 8.47 | 200 | 25 | 5.26 | 8 | . 373 |
| $1 A^{\prime \prime}$ | . 53 | 5.98 | 11.27 | . 316 | 9.46 | 100 | 60 | 11.11 | 28 | . 382 |
| 2 A | . 51 | 4.97 | 9.75 | . 476 | 5.22 | 50 | 35 | 5.15 | 7 | . 362 |
| $2 B$ | . 51 | 4.94 | 9.69 | . 482 | 5.12 | 30.3 | 50 | 8.23 | 24 | . 396 |
| 20 | . 50 | 4.90 | 9.80 | . 500 | 4.89 | 21.8 | 50 | 5.26 | 30 | . 415 |
| 3A | . 48 | 3.72 | 7.75 | . 893 | 2.06 | 15.9 | 30 | 6.39 | 25 | 12 |
| 3B | . 47 | 3.70 | 7.87 | . 933 | 1.99 | 11.0 | 40 | 5.82 | 28 | . 433 |
| 3 C | . 48 | 3.71 | 7.73 | . 908 | 2.04 | 6.9 | 50 | 6.05 | 33 | . 425 |
| 4 A | . 71 | 5.00 | 7.04 | . 338 | 7.40 | 71.7 | 15 | 7.07 | 9 | 405 |
| 4 B | . 72 | 5.00 | 6.94 | . 333 | 7.50 | 33.3 | 40 | 8.94 | 18 |  |
| 40 | . 73 | 4.99 | 6.84 | . 330 | 7.55 | 25.0 | 50 | 8.70 | 12 | . 392 |
| 5A | . 7 | 3. | 5.32 | . 503 | 3.97 | 9.1 | 20 | 6.84 | 30 | 887 |
| 5B | . 74 | 3.98 | 5.38 | . 512 | 3.89 | 10.5 | 40 | 6.61 | 32 | . 380 |
| 5 C | . 75 | 3.98 | 5.31 | . 505 | 3.94 | 8.3 | 40 | 6.61 | 39 | . 384 |
| 6 6. | . 7 | 2.92 | 3.90 | . 936 | 1.56 | 10.5 | 30 | 11.11 | 40 | . 384 |
| 6 B | . 75 | 2.95 | 3.93 | . 917 | 1.61 | 50.0 | 45 | 6.38 | 40 | . 39 |
| 6 C | . 74 | 2.91 | 3.94 | . 957 | 1.52 | 8.0 | 50 | 6.83 | 28 | . 402 |
| 7 A | . 74 | 2.00 | 2.70 | 2.03 | . 494 | 33.3 | 40 | 6.38 | 7 | 36 |
| 7 B | . 75 | 2.01 | 2.68 | 1.99 | . 506 | 18.2 | 60 | 13.62 | 14 | 452 |
| 70 | . 76 | 2.03 | 2.67 | 1.92 | . 528 | 100.0 | 60 | 11.72 | 28 | 418 |
| 8 A | . 35 | 5.88 | 16.8 | . 497 |  | 66.7 | 30 | 5.26 | 10 | .391 |
| 8B | . 35 | 5.90 | 16.8 | . 492 | 6.01 | 66.7 | 30 | 5.26 | 9 | 38 |
| 8 C | . 3 | 5.89 | 16.8 | , | 5.95 | 200.0 | 30 | 5.54 | 10 | . 39 |
| 9 A | . 37 | 3.00 | 8.12 | 1.80 | . 833 | 200.0 | 25 | 6.95 | 18 | . 391 |
| 9 B | . 40 | 3.00 | 7.70 | 1.67 | . 900 | 100.0 | 30 | 6.95 | 21 | . 391 |
| 9 C | . 38 | 3.00 | 8.01 | 1.78 | . 843 | 1 67.0 | 25 | 6.95 | 22 | 40 |

Symbol
b Breadth of specimen inches
h Depth of specimen inches
h/b Depth-breadth ratio
y/I Section modulus -
I Moment of inertia of

## Symbol Significance

Slope Number of inches for l-inch rise of grain \%SG Percent summer growth Percent moisture Rate of growth - rings per inch
Specific gravity section - (inches) ${ }^{4}$

Table II.
Single Load Tests

|  | Failure |  | Apparent |  | $\mathrm{h} / \mathrm{b}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| specimen | Load Manner |  | E | $\pm$ |  |
| 1 A : | 1660 | lat. | 1062 | 6140 | 11.32 |
| $10^{1}$ | 1450 | lat. | 873 | 5920 | 11.76 |
| $1 \mathrm{~A}^{\text {P }}$ | 1925 | lat. | 1043 | 71.50 | 11.27 |
| 2A | 1060 | lat. | 1220 | 5900 | 9.75 |
| 2B | 1515 | lat. | 1310 | 8600 | 9.69 |
| 20 | 1565 | lat. | 1230 | 9200 | 9.80 |
|  | 830 | lat. | 1330 | 8700 | 7.75 |
| 3B | 830 | lat. | 1280 | 9100 | 7.87 |
| 30 | 770 | lat. | 1220 | 8200 | 7.73 |
| 4 A | 2240 | ten. | 1360 | 8900 | 7.04 |
| 4B | 2360 | com. | 1280 | 9300 | 6.94 |
| 4 C | 2320 | com. | 1440 | 9000 | 6.84 |
| 5 A | 1580 | ten. | 1190 | 9300 | 5.32 |
| 5B | 1500 | ten. | 1300 | 9050 | 5.38 |
| 50 | 1660 | ten. | 1240 | 9850 | 5.31 |
| 6A | 870 | ten. | 1470 | 9570 | 3.90 |
| 6 B | 690 | ten. | 1342 | 7420 | 3.93 |
| 60 | 930 | ten. | 1356 | 10410 | 3.94 |
|  |  | com. | 1570 | 10730 | 2.70 |
| 7 B | 460 | com. | 1990 | 10760 | 2.68 |
| 70 | 420 | com. | 1310 | 9490 | 2.67 |
|  | 600 | lat. | 1220 | 3500 | 16.8 |
| 8 B | 510 | lat. | 1125 | 2950 | 16.8 |
| 80 | 720 | lat. | 1383 | 4190 | 16.8 |

Load is maximum scale reading in pounds.
Lat. signifies lateral failure.
com. " compression failure.
ten. " tension failure.
$E$ is Modulus of Elasticity calculated from plot made as the specimen was loaded - pounds/square-inch.
$f$ is apparent modulus of rupture figured from the load given here - pounds/square-inch.
$h / b$ is the depth-breadth ratio of the specimen.

Table III.
Original Test Data
Span Tests (Single Load)
Failure

| Span | Failure Load Manner |  | $\begin{aligned} & \text { Appa } \\ & \mathrm{E} / 1000 \end{aligned}$ | f | $\mathrm{h} / \mathrm{b}$ | Specimen |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 57 | 230 | lat. | 1830 | 5910 | 8.12 | 9 A |
| 57 | 260 | lat. | 1750 | 6180 | 7.50 | 9 B |
| 57 | 245 | lat. | 2055 | 6210 | 8.01 | 90 |
| 51 | 270 | lat. |  | 6200 |  | 9A |
| 51 | 290 | lat. |  | 6170 |  | 93 |
| 51 | 290 | lat. |  | 6580 |  | 90 |
| 45 | 370 | lat. |  | 7480 |  | 9 A |
| 45 | 440 | lat. |  | 8250 |  | 9 B |
| 45 | 375 | lat. |  | 7510 |  | 90 |
| 40 | 440 | lat. |  | 7920 |  | 9 A |
| 40 | 520 | lat. |  | 8670 |  | 9 B |
| 40 | 490 | lat. |  | 8720 |  | 90 |
| 35 | 570 | lat. |  | 8970 |  | 9A |
| 35 | 680 | lat. |  | 9920 |  | 9 B |
| 35 | 600 | lat. |  | 9360 |  | 90 |
| 30 | 810 | lat. |  | 10930 |  | 9 A |
| 30 | 960 | lat. |  | 12000 |  | 93 |
| 30 | 910 | lat. |  | 12150 |  | 90 |
| 25 | 970 | com. |  | 10900 |  | 9 9 |
| 25 | 1120 | com. |  | 11670 |  | 9 B |
| 25 | 1025 | com. |  | 11400 |  | 90 |

Load is the maximum scale reading in pounds.
Lateral failure is signified by lat.
Compression
$E$ is modulus of elasticity in pounds per square-inch calculated from plot made as the specimen was loaded with 57-inch span.
$f$ is apparent modulus of rupture, figured from the load given here - pounds/square-inch.
$h / b$ is the depth-breadth ratio of the specimen.

Table IV.

## Two Point Loading Tests

| Spec- <br> imen | Failure <br> Ioad Manner |  | ${\underset{f}{\text { Apparen }}}^{\text {far }}$ | $\mathrm{h} / \mathrm{b}$ | Span | a | Chips |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 8A | 910 | lat. | 3540 | 16.8 | 47 | 15.67 |  |
| 8 B | 950 | lat. | 3660 | 16.8 | 47 | 15.67 |  |
| 8 C | 950 | lat. | 3690 | 16.8 | 47 | 15.67 |  |
| 3 A | 1065 | lat. | 7450 | 7.75 | 4.7 | 15.67 |  |
| 3 B | 1160 | lat. | 8490 | 7.87 | 47 | 15.67 |  |
| 3 c | $870^{\circ}$ | * | 6200 | 7.73 | 47 | 15.67 |  |
| 2 A | 1770 | lat. | 6600 | 9.75 | 47 | 15.67 | 1-1 |
| $1 A^{\prime}$ | 3380 | lat. | 7540 | 11.32 | 44 | 14.17 | 3-3 |
| $10^{1}$ | 2560 | ton. | 6300 | 11.76 | 44 | 14.17 | 3-3 |

* tension at a knot.

Load is the maximum scale reading in pounds.
lat. signifies lateral failure.
ton.
$f$ is tho apparont modulus of rupture figured from the loads given here - pounds/square-inch.
$h / b$ is the depth-breadth ratio.
Chips noted are the ones uscd to prevent orushing at the supports.
a is the arm used in computing the moment in calculating the modulus of rupture - inches.


Table V.
Dopth Breadth Tests Corrected Values

| Specimen | $\mathrm{f}_{0}$ | $M_{C}$ | $\mathrm{MC}^{\prime}$ | $\mathrm{E}_{\mathrm{c}}$ | $\mathrm{h} / \mathrm{b}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $1 A^{\prime}$ | 6970 | 22100 | 15000 | 1173000 | 11.32 |
| $10^{\prime}$ | 5800 | 16700 | 12800 | 858000 | 11.76 |
| $1 A^{\prime \prime}$ | 8660 | 27400 | 18500 | 1235000 | 11.27 |
| 2 A | 5770 | 12100 | 11600 | 1234000 | 9.75 |
| 2 B | 9210 | 18700 | 18000 | 1345000 | 9.69 |
| 2 C | 8620 | 17200 | 17400 | 1219000 | 9.80 |
| 3 A | 9010 | 10100 | 14900 | 1358000 | 7.75 |
| 3 B | 9380 | 10100 | 15500 | 1397000 | 7.87 |
| 30 | 10880 | 12000 | 17800 | 1621000 | 7.73 |
| 4 A | 8550 | 25300 | 14600 | 1345000 | 7.04 |
| 4 B | 10350 | 30900 | 17500 | 1441000 | 6.94 |
| 4 C | 10060 | 30500 | 17000 | 1508000 | 6.84 |
| 5A | 11710 | 23300 | 17400 | 1556000 | 5.32 |
| 5B | 11170 | 21900 | 16700 | 1628000 | 5.38 |
| 50 | 12660 | 25100 | 18800 | 1675000 | 5.31 |
| 6 A | 12970 | 13400 | 15500 | 1966000 | 3.90 |
| 6.3 | 7230 | 7900 | 9000 | 1321000 | 3.93 |
| 60 | 12860 | 13400 | 15800 | 955000 | 3.94 |
| 7 A | 11400 | 5600 | 10400 | 1677000 | 2.70 |
| 73 | 11960 | 6000 | 10900 | 1757000 | 2.68 |
| 70 | 10190 | 5300 | 9400 | 1417000 | 2.67 |
| 8 A | 2880 | 5800 | 7200 | 1151000 | 16.8 |
| 83 | 2500 | 5100 | 6300 | 1074000 | 16.8 |
| 80 | 3450 | 7000 | 8700 | 1301000 | 16.8 |

$f_{c}$ is the corrected modulus of rupture in pounds per square-inch, the sum of the apparent modulus of rupture from Table II and the corrections.
$\mathbb{M}_{C}$ is the maximum bending moment calculated from $f_{C}$ in pouna inches.
$\mathbb{M}_{c}^{\prime}$ is $\mathbb{M}_{c}$ corrected to a constant sectional area of 2.46 square inches, in pound-inches.
$E_{C}$ is the corrected modulus of elasticity.
$\mathrm{h} / \mathrm{b}$ is the depth-brcadth ratio.

Table VI.
Span Tests Corrected Values

| Specimen | $\mathrm{f}_{\mathrm{C}}$ | $M_{C}$ | $\mathrm{M}_{6} \mathrm{C}^{1}$ | $\mathrm{E}_{\mathrm{C}}$ | Span | $\mathrm{h} / \mathrm{b}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $\begin{aligned} & 9 A \\ & 9 B \\ & 9 C \end{aligned}$ | $\begin{aligned} & 5900 \\ & 6170 \\ & 5760 \end{aligned}$ | $\begin{aligned} & 3280 \\ & 3700 \\ & 3230 \end{aligned}$ | $\begin{array}{r} 10070 \\ 10700 \\ 9790 \end{array}$ | $\begin{aligned} & 1799000 \\ & 1719000 \\ & 2072000 \end{aligned}$ | $\begin{aligned} & 57 \\ & 57 \\ & 57 \end{aligned}$ | $\begin{aligned} & 8.12 \\ & 7.50 \\ & 8.01 \end{aligned}$ |
| $\begin{aligned} & 9 \mathrm{~A} \\ & 9 \mathrm{~B} \\ & 9 \mathrm{C} \end{aligned}$ | $\begin{aligned} & 6190 \\ & 6160 \\ & 6130 \end{aligned}$ | $\begin{aligned} & 3440 \\ & 3690 \\ & 3440 \end{aligned}$ | $\begin{aligned} & 10550 \\ & 10650 \\ & 10470 \end{aligned}$ |  | $\begin{aligned} & 51 \\ & 51 \\ & 51 \end{aligned}$ |  |
| $\begin{aligned} & 9 A \\ & 9 B \\ & 9 \mathrm{C} \end{aligned}$ | $\begin{array}{r} 7470 \\ -8240 \\ 7060 \end{array}$ | $\begin{aligned} & 4150 \\ & 4933 \\ & 3960 \end{aligned}$ | $\begin{aligned} & 12720 \\ & 14250 \\ & 12000 \end{aligned}$ |  | $\begin{aligned} & 45 \\ & 45 \\ & 45 \end{aligned}$ |  |
| $\begin{aligned} & 9 \mathrm{~A} \\ & 9 \mathrm{~B} \\ & 9 \mathrm{C} \end{aligned}$ | $\begin{aligned} & 7910 \\ & 8660 \\ & 8270 \end{aligned}$ | $\begin{aligned} & 4390 \\ & 5190 \\ & 4640 \end{aligned}$ | $\begin{aligned} & 13500 \\ & 15000 \\ & 14070 \end{aligned}$ |  | $\begin{aligned} & 40 \\ & 40 \\ & 40 \end{aligned}$ |  |
| $\begin{aligned} & 9 A \\ & 9 B \\ & 9 \mathrm{C} \end{aligned}$ | $\begin{aligned} & 8960 \\ & 9910 \\ & 8910 \end{aligned}$ | $\begin{aligned} & 4980 \\ & 5940 \\ & 5000 \end{aligned}$ | $\begin{aligned} & 15300 \\ & 17150 \\ & 15150 \end{aligned}$ |  | $\begin{aligned} & 35 \\ & 35 \\ & 35 \end{aligned}$ |  |
| $\begin{aligned} & 9 A \\ & 9 B \\ & 9 C \end{aligned}$ | $\begin{aligned} & 10920 \\ & 11990 \\ & 11700 \end{aligned}$ | $\begin{aligned} & 6070 \\ & 7180 \\ & 6570 \end{aligned}$ | $\begin{aligned} & 18600 \\ & 20700 \\ & 20200 \end{aligned}$ |  | 30 30 30 |  |
| $\begin{aligned} & 9 A \\ & 9 B \\ & 90 \end{aligned}$ | $\begin{aligned} & 10890 \\ & 11660 \\ & 10950 \end{aligned}$ | $\begin{aligned} & 6050 \\ & 6980 \\ & 6150 \end{aligned}$ | $\begin{aligned} & 18600 \\ & 180150 \\ & 18650 \end{aligned}$ |  | $\begin{aligned} & 25 \\ & 25 \\ & 25 \end{aligned}$ |  |

$f_{C}$ is the corrected modulus of rupture in pounds per squareinch, the sum of the apparent modulus of rupture from Table III and the corrections.
$M_{C}$ is the maximum bending moment in pound-inches calculated from $f_{c}$.
$M_{C}{ }^{1}$ is $M_{C}$ corrected to a constant sectional area of 2.46 square-inches, in pound-inches.
$E_{C}$ is the corrected modulus of elasticity.
$h / b$ is the depth-breadth ratio.

Table VII.

$f_{c}$ is the corrected modulus of mature in pounds per square-inch, the sum of the apparent modulus of mature from Table IV and the corrections.
$\mathbb{M}_{C}$ is the maximum bending moment in pound-inches calcurated from $f_{C}$.
$M_{C}$ ' is $M_{C}$ corrected to a constant sectional area of 2.46 square-inches, in pound-inches.
$h / b$ is the depth-breadth ratio.

## References

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Fig. 4 Depth-breadth ratio vs. corrected modulus of rurture.


Fig. 5 Span vs.corrected modulus of rupture.


Fig. 6 Depth-breadth ratio va. corrected modulus of rupture.


Fig. 7 Depth-breadth ratio vs corrected maximum mornent.


Fig. 8 Span vs. corrected maximum moment.


Fig. $\theta$ Depth-breadth ratio vs. corrected modulus of elasticity.


Fig. 10 Deoth-oreadth ratio vs. span-breadth ratio.

