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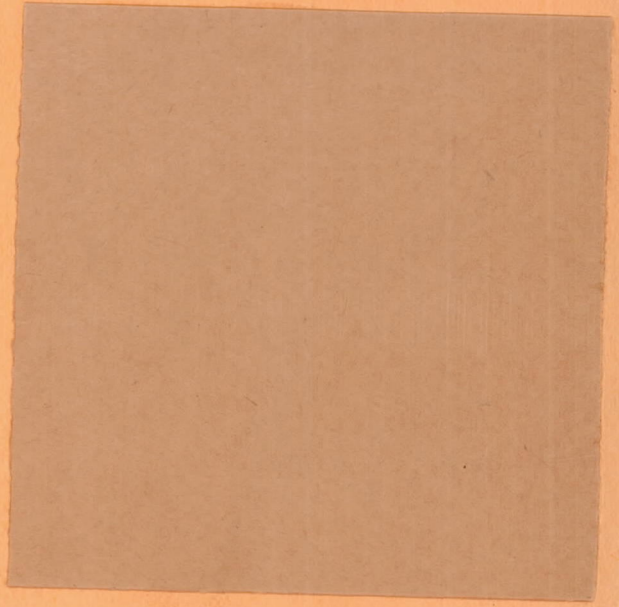
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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS  
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THE DRAG OF AIRSHIPS

DRAG OF BARE HULLS - II

By Lieut. Clinton H. Havill, U.S.N.



Washington  
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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS.

TECHNICAL NOTE NO. 243.

THE DRAG OF AIRSHIPS.

DRAG OF BARE HULLS - II.

By Clinton H. Havill.

Summary

The extension of wind tunnel tests of models of airship hulls to full scale requires an extension from a  $VL$  of the order of less than 500 sq.ft./sec., to the order of 80000 sq.ft./sec., where  $V$  = air speed, feet per second,  $L$  = length in feet of the particular form of hull. The reason for this research was to furnish the airship designer with a method for finding the  $VL$  curve of any conventional type of hull, using data obtained from actual performance of airships flown prior to 1926.

This digest as given here in Part II, was begun in preliminary details, in June, 1922, and completed in April, 1926, as it was necessary to complete Part I before Part II could be completed; the period between September, 1923, and December, 1925, was devoted to work on Part I.

The outstanding results are as follows:

1. An empirical method for finding the drag coefficient of any bare airship hull with its  $VL$  curve from 100,000 cu.ft. volume to 6,400,000 cu.ft. volume. (See diagrams Figs. 7 and 8

and example to illustrate its use.)

2. The derivation of an empirical shape coefficient that can be calculated from the hull contour that defines the VL curve of any conventional airship shape within the limits placed on Figs. 7 and 8.

3. (a) That the slope of each VL curve differs with each type of hull and that its slope is not quite constant.

(b) That  $C_H = \text{function of } (VL)^n$  and  $n$  is a variable at different values of VL.  $C_H = \text{drag coefficient of bare airship hull. Drag} = C_H \frac{\rho}{2} (\text{Volume})^{2/3} v^2.$

(c) That the value of  $n$  varies slowly so that extrapolations beyond that given by diagrams Figs. 7 and 8 of the VL curve are not much in error, as requirement 3 of illustrative problem shows.

4. The region from model tests to a volume of 100,000 cu.ft. size indicates that in this region the most rapid change in the slope occurs with the conclusion that "The best model in the wind tunnel will probably be the best (lowest drag) airship hull but not necessarily" as their VL curves may cross and again may re-cross at higher values of VL. In view of this as found by extrapolating the VL curves calibrated on performance back to wind tunnel values and extrapolating wind tunnel results to higher values of VL together with the fact that airship designers are not interested in airship hulls of less than

100,000 cu.ft. of volume, this part of these researches was left out. The scale on diagrams at .3 cu.ft. volume calibrated on existing wind tunnel data is merely for general information.

### Introduction

The principal components of the drag of bodies in a wind stream has been laid down by Reynolds, Stanton, Munk, Prandtl, Froude, Bairstow and others, so that it is not necessary to outline their work here. Reference to the summary of their work in the recent N.A.C.A. Technical Report No. 219, "Some Aspects of the Comparison of Model and Full Scale Tests" by D. W. Taylor, is invited, which expressed in words: Drag = pressure difference + skin friction + wave making + compressibility effect.

$$\text{Symbols: Drag} = R = \text{Drag} = F_1 (\rho L^2 V^2) F_2 \left( \frac{\rho VL}{\mu} \right) F_3 \left( \frac{Lg}{V^2} \right) F_4 \left( \frac{V^2}{V_s^2} \right)$$

L = Linear dimensions of length.

V = Air speed.

$\rho$  = Mass density of air.

$\mu$  = Viscosity.

$V_s$  = Velocity of sound in air.

G = Acceleration of gravity.

R = Drag.

It has been well established in theory and practice that as far as airships are concerned the compressibility effect expressed by  $\left( \frac{V^2}{V_s^2} \right)$  is negligible or zero as the air speeds in flight are

so far below the speed of sound at which compressibility exists. The wave making  $\left(\frac{Lg}{V^2}\right)$  so important in surface ships is negligible in airships and if it does exist in a microscopic percentage, can be included in the constants and exponents in the remaining two. So that  $R = F_1 (\rho L^2 V^2) F_2 \left(\frac{\rho VL}{\mu}\right)^n$  where  $n$  is a variable depending on type of hull - fineness ratio, virtual volume, length, diameter, eccentricity of nose ellipse, cylindrical coefficient, and on the value of  $VL$  as found out in this research. Or, if reduced to a standard value of kinematic viscosity of  $\frac{\rho}{\mu}$  then  $R = \text{constant} (\rho L^2 V^2) F_2 (VL)^n$ .

Let  $2K = \text{the constant}$ ;  $(\text{Volume})^{2/3} = L^2$ ,

then  $R = K \frac{\rho}{2} (\text{Volume})^{2/3} V^2, F_2 (VL)^n$ .

Let  $C_H = K + \frac{F_2 (VL)^n}{\frac{\rho}{2} (\text{Volume})^{2/3} V^2}$ ,

then  $R = C_H \frac{\rho}{2} (\text{Volume})^{2/3} V^2$  in which case it is seen that  $C_H$  is a variable depending on the value of  $(VL)^n$ .

It now remains to give a method of finding the value of  $C_H$  knowing the contour and size of the airship hull. In brief, this was done by taking the whole ship performance of a large number of ships (all Zeppelin types and Navy nonrigids) as given in Part I, and calculating their external drag and getting the hull drag. Then to find a quantity of linear dimensions that is calculated from the contour and size of each ship such that

if the drag is plotted against this VL that the results show it to be a smooth curve. With this as a basis, it now was necessary to find a dimensionless quantity that would define each ship - such a quantity called here "whole hull shape coefficient" (Y + Z) such that it could be calibrated against the various values of  $C_H$  based on performance.

### Body of Report

An exhaustive research was made to find a dimensionless quantity that sufficiently defines a given hull and to express the relation between  $C_H$  at various values of VL and this quantity. The effective velocity over the skin of different types of hulls at different speeds was found to be so different that it could not be expressed as a constant times air speed, so the surface area times  $KV^n$  was given up as n apparently was a very sensitive quantity. So shapes were geometrically expanded to the volume of known ships for comparison. From this comparison, relative drag coefficients were obtained by discovering that the drag of an airship hull follows very closely the VL principle over a short range and results are comparable if L is defined as  $L_g$  defined here as geometric length where

$$L_g = \sqrt[4]{(\text{Volume}) + \frac{\pi L^3}{3}} \quad (\text{length}) = \sqrt[4]{(\text{Virtual Vol.})} \quad (\text{length});$$

this was discovered by trial and error in analyzing the wind tun-

nel results and plotting their drag in pounds versus  $VL_g$  as shown in Fig. 6.

The external drags of all the items (about 90 hulls - 26 separate types) of Part I can be separated by calculating the external drags of about six types of hulls and by simultaneous equations solving for the external drags of all the remaining types of hulls. However, the results are no better than the correctness of the external drag of the five or six types calculated. Yet these results when plotted against  $VL_g$  show a smooth curve. For this report it was better, therefore, to calculate the external drag for all the 26 types of hulls (given in Part I) and to plot them against  $VL_g$  (Fig. 1) is such a curve.

There is another way in which the external drag of various airships can be calculated, and that is to assume that the percentage of external drag remains the same part of the total as wind tunnel experiments indicate. In general, wind tunnel results show nonrigid types to have about 60% total drag = external drag; and rigid Zeppelin types to have 40% total drag = external drag. The exact percentage will of course vary with the type of cars, fins, struts, wires, etc., but various percentages can be assumed on each type based entirely on engineering judgment. The remaining hull drags, if plotted against  $VL_g$ , will give Fig. 2.

Now the mean between Fig. 1 and Fig. 2, is Fig. 3. In view of the fact that Fig. 1 and Fig. 2 give a curve that is practi-

cally identical, it gives in Fig. 3 a basis of comparison of hull drag coefficients when ships are expanded or contracted to the same volume and the same speed. In other words, the ratio of hull drag coefficients ( $C_H$ ) at the same volume and speed is the ratio of the drags of the bare hulls as

$$\frac{\text{Drag of hull 1}}{\text{Drag of hull 2}} = \frac{C_{H1} \frac{\rho}{2} (\text{Vol})^{2/3} V^2}{C_{H2} \frac{\rho}{2} (\text{Vol})^{2/3} V^2};$$

if  $\rho$ , (Vol), and  $V$  are the same for both ships, then

$$\frac{\text{Drag of hull 1}}{\text{Drag of hull 2}} = \frac{C_{H1}}{C_{H2}}.$$

Now with curve [drags, vs.  $(VL_g)$ ] as in Fig. 3, the comparison of ships at different volumes and  $V = 100$  ft./sec., can be carried out. A comparison at 100,000; 200,000; 400,000; 800,000; 6,400,000 was carried out. It necessitated a small extrapolation of curve (Fig. 3) to get 6,400,000 yet as the curve is fairly definite and the value of  $\left(\frac{\rho}{\mu} VL_g\right)^n$  shows  $n$  to change value so slowly that this extrapolation is justified.

From here on various methods were tried to find a dimensionless quantity which would show to be a function of these values of  $C_H$  that comparison indicated. If such a quantity was established it could be represented on a plot or diagram and calibrated on the comparative results.

Speed and density was kept constant so that for a given volume the relative values of  $C_H$  were the same as the relative



values of their drags as  $\frac{\rho}{2} (\text{Vol})^{2/3} V^2 = \text{constant}$ . The dimensionless quantity that proved to sufficiently define a hull and to have no conflicts with the comparative results was  $(Y + Z)$ .  
 $Y = (\text{eccentricity of nose ellipse}) (\text{cylindrical coefficient})$   
 $(\text{fineness ratio}); Z = \left( \frac{\text{length}}{\text{geometric length}} \right) (\text{fineness ratio}).$

Hulls were now grouped according to their values of  $Y$  and the parametric equation of  $Y$  against  $C_H$  was plotted (Fig. 4) where  $C_H$  was the total hull drag coefficient of ships with the same value of  $Y$ . A mean curve was drawn through the points plotted - a curve for volumes 100,000; 800,000; 6,400,000 cu.ft. Likewise, for  $Z$  on Fig. 5. It is to be noted that

$$Y = (e) \left( \frac{4 \text{ Vol}}{\pi D^2 L} \right) \times \left( \frac{L}{D} \right) = (e) \left( \frac{4 \text{ Vol}}{\pi D^3} \right),$$

is independent of length except as length affects volume. An interesting research by simultaneous equations by the author reveals that this function  $Y$ , for the ten ships on which it was calculated, appears to be a true function of that part of the drag due to pressure difference, and that  $KYL^2V^2 + F_2 Z \left( \frac{\rho VL}{\mu} \right)^n = R$  gives  $K$  a constant for all values of  $VL$ . The writer hopes to be able to analyze all existing ships, in the near future, in order to prove or disprove this relation. Rather letting Fig. 4 indicate

$C_H = F_1 Y + F_2 Z$  and plot total  $C_H$  against  $Y$  and likewise  $Z$  in Fig. 5. This amounts to a calibration of  $Y$  and  $Z$  on

$$C_H. \quad Z = \frac{L}{L_g D} \times \frac{L}{D} = \frac{L^2}{L_g D} \quad \text{gives length the predominate factor}$$

effect in Z. Now with the values of Y and Z for each model in the wind tunnel the values of  $C_H$  according to Y called  $C_{HY}$  and the values of  $C_H$  called  $C_{HZ}$  according to Z were picked off. To let each have its proper effect, the formula

$$\frac{Y C_{HY} + Z C_{HZ}}{Y + Z} = C_H \text{ for given } (Y + Z) \text{ was used to give the}$$

value of  $C_H$  at the various volumes. With these various values of  $C_H$  from model to full scale on the 17 models, the scales could be calibrated.

The interval from .3 cu.ft. volume to 100,000 cu.ft. volume was calibrated on the diagrams (Figs. 7 and 8) and the slope given. The remaining ships from Part I were now added to give a complete calibration at 100,000; 800,000; and 6,400,000 cu.ft. volume; (An exploration of the region just beyond the usual wind tunnel model size (100 cu.ft. volume) indicates that perhaps some very sharp changes in VL curve is probable) so that the slope lines from .3 cu.ft. to 100,000 cu.ft. are the mean over this part of the VL curve. However, beyond 100,000 cu.ft. volume the diagrams in Figs. 7 and 8 will give the VL curve very accurately if used in the manner as shown by the example (Fig. 9). Since the scales are not uniform sight interpolation of values of  $C_H$  at various volumes other than 100,000; 800,000; and 6,400,000 are very misleading. The illustrative problem shows how to get the value of  $C_H$  (from the VL curve obtained) for other volumes.

The limits from which this data is designed are placed on each diagram and there is no justification for using it other than within the limits given. However, these limits will cover practically all contours of airship hulls that exist or are proposed today.

Further ground for research is to separate bare hull drag into pressure difference and skin friction, a large part of which has been done during the trial and error methods used to discover the quantities  $Y$  and  $Z$ .

#### Assumptions

1. That external drag, cars, fins, wires, etc., vary as the square of the speed.
2. The coefficients used in calculating drag of cars, fins, etc., were assumed based on engineering judgment. The idea was to get the curve drag versus  $(VL_g)$  oriented at the proper order of magnitude as a further check on the results which would be obtained by the percentage of external drag method. However, it is believed that the coefficients used to calculate drag of cars, fins, wires, etc., are as nearly correct as the present science of aerodynamics can give.

Units used throughout this report are ft., lb., sec.

Everything in this report is reduced to:

A standard density of  $\rho = .00237$  slugs/cu.ft.

A standard viscosity of  $\mu = .0000003779$  slugs/ft.sec.

A standard kinematic viscosity of  $\nu = \frac{\mu}{\rho} = .000159$  sq.ft./  
sec.

Example to Illustrate Method for Use of Diagrams, Figs. 7 & 8.  
Part II.

Problem:

An Airship hull is constructed with a contour like the U.S.S. Los Angeles if 100 ft. parallel section had been added at the point of maximum ordinate; and to make it such dimensions that the air volume of hull = 5,000,000 cu.ft.

Required:

1. Hull drag coefficient- $V_s$ - $V_L$ , curve of this airship hull
2. Bare hull drag in lb. at 100 ft./sec., standard density, ( $\rho = .00237$  slugs/cu.ft.)
3. Horsepower absorbed in overcoming bare hull drag at 120 ft./sec.

Data

Present dimensions of U.S.S. Los Angeles.

			Symbol
Air volume of hull	2,764,461.0	cu.ft.	(Vol)
Length	658.3	ft.	L
Maximum diameter	90.7	ft.	D
Cylindrical coefficient	.650		$\left(\frac{4(\text{Vol})}{\pi D^2 L}\right)$
Eccentricity of nose ellipse	.978		$\left(\frac{\sqrt{x^2 - r^2}}{x}\right) = e$
Fineness ratio	7.25		L/D

Calculations of dimensions of hull in problem and dimensionless quantities of shape.

(Vol) added in parallel section  $100 \pi r^2 = 100 \pi 45.35^2 = 646,103$  cu.ft.

Former (Vol) 2,764,461  
 New (Vol) 3,410,569

New length =  $L + 100 = 658.3 + 100 = 758.3$  ft.

Max. diameter, as formerly 90.7 ft.

e .978

New fineness ratio  $\frac{L}{D} = \frac{758.3}{90.7} = 8.36$

Cylindrical coef. =  $\frac{(\text{Vol})}{\text{Vol of circumscribing cylinder}} = \frac{4(\text{Vol})}{\pi D^2 L} =$   
 $= \frac{4 \times 3,410,569}{\pi(90.7)^2 \times 758.3} = \frac{13,642,276}{19,597,764} = .6961$

Virtual (Vol) =  $V_M (\text{Vol}) + \frac{\pi r^3}{3} = 3,410,569 + \frac{\pi(45.35)^3}{3} =$   
 $= 3,410,569 + 97,670 = 3,508,239$  cu.ft.

$\text{Log}_{10} V_M = 6.54509$

$\text{Log}_{10} L = 2.87984$

"  $(V_M L) = 9.42493$

$L_g = \text{Geometric length} =$

$= \sqrt[4]{(\text{Vol} + \frac{\pi r^3}{3}) (\text{length})} =$

$= \sqrt[4]{V_M L} = 227.1 \text{ ft.}$

"  $\sqrt[4]{V_M L} = \frac{9.42493}{4} = 2.35623 = \log_{10} L_g; L_g = 227.1 \text{ ft. at}$   
 $\text{Vol} = 3,410,569 \text{ cu.ft.}$

$Y = (e) (\text{cylindrical coef.}) (\text{fineness ratio})$

$Y = .978 \times .6961 \times 8.36 = 5.691$

$Z = \frac{L (\text{fineness ratio})}{L_g} = \frac{L^2}{L_g D} = \frac{(758.3)^2}{227.1 \times 90.7} = \frac{575,020}{20,598} = 27.916$

$(Y + Z) = 5.691 + 27.916 = 33.607$

$\text{Log}_{10} (Y + Z) = 1.52643$

Note:

[e,  $\frac{L}{D}$ ,  $\frac{4 \text{ Vol}}{\pi D^2 L}$ ,  $\frac{L}{L_g}$ , are dimensionless quantities and can be calculated from any set of dimensions that pertain to the same volume. (Y + Z) - independent of volume.]

When  $L = 758.3 \text{ ft.}$        $\text{Vol.} = 3,410,569 \text{ cu.ft.}$

$\left(\frac{L \text{ at } 100000}{758.3}\right)^3 = \frac{100000}{3,410,569} = .02932$

$L \text{ at } 100000 = \sqrt[3]{(758.3)^3 \times .02932} = \sqrt[3]{12,785,000} = 233.83 \text{ ft.}$

By  $\text{logs}_{10}$

$\log 758.3^3 = 3 \times 2.87984 = 8.63952$

"  $.02932$

$\frac{8.46716-10}{3} = 7.10668$

$\log L \text{ at } 100000$

$2.33889$

$L \text{ at } 100000 = 233.83 \text{ ft.}$

$\frac{\text{Desired Vol}}{100000} = \frac{5,000,000}{100000} = 50.$

log (L at 100,000) <sup>3</sup>	as before	7.10688
" 50		<u>1.69897</u>
		3   <u>8.80585</u>
log L <sub>5,000,000</sub>		3.93522 = 861.41 ft. = length of desired hull.

## Requirement 1:

$$L \text{ at } 100,000 \text{ ft.}^3 = 233.83 \text{ ft.}$$

$$L \text{ at } 800,000 \text{ ft.}^3 = 467.66 \text{ ft.}$$

$$L \text{ at } 6,400,000 \text{ ft.}^3 = 935.32 \text{ ft.}$$

$$L \text{ at } 5,000,000 \text{ ft.}^3 = 861.41 \text{ ft.}$$

$$VL \text{ at } 100,000 \text{ \& } 100 \text{ ft./sec.} = 100 \times 233.83 = 23383 \text{ ft.}^2/\text{sec.} \quad \text{Log}_{10} VL = 4.36889$$

$$VL \text{ at } 800,000 \text{ \& } 100 \text{ ft./sec.} = 100 \times 467.66 = 46766 \text{ ft.}^2/\text{sec.} \quad \text{Log}_{10} VL = 4.66992$$

$$VL \text{ at } 6,400,000 \text{ \& } 100 \text{ ft./sec.} = 100 \times 935.32 = 93532 \text{ ft.}^2/\text{sec.} \quad \text{Log}_{10} VL = 4.97095$$

$$VL \text{ at } 5,000,000 \text{ \& } 100 \text{ ft./sec.} = 100 \times 861.41 = 86141 \text{ ft.}^2/\text{sec.} \quad \text{Log}_{10} VL = 4.93522$$

Enter left-hand scale of Fig. 8 with  $\log_{10} (Y + Z) = 1.52643$  and follow across to scale .3 cu.ft. Vol. (see dotted line, Fig. 8).

From .3 cu.ft. Vol., interpolate for slope and follow across to 100,000 cu.ft. scale (see dotted line).

From 100,000 cu.ft. scale, follow across, interpolating for slope, to 800,000 and 6,400,000 cu.ft. scales.

From 800,000 to 6,400,000 scale is a straight line (see dotted line solution of this problem in Fig. 8).

Pick off the following values of  $C_H$ , and take logs:

Volume	$C_H$	$\log_{10} C_H$
100,000	.02180	8.33846-10
800,000	.01654	8.21854-10
6,400,000	.01380	8.13988-10

Note: Figs. 7 and 8 are for a speed of 100 ft./sec.,

$\rho = .00237$  slugs/ft.<sup>3</sup>, and standard  $\rho/\mu$ . Enter Fig. 9 with  $\log_{10} VL = 4.93522$  and from curve pick off

$\log_{10} C_H = 8.147-10$ . Whence  $C_H = .01403$  at 5,000,000

cu.ft. and 100 ft./sec. Use this value in Requirement 2.

Requirement 2:

Bare hull drag at 100 ft./sec.  $\rho = .00237$  slugs/cu.ft.

$$\text{Drag} = C_H \frac{\rho}{2} (\text{Vol})^{2/3} V^2$$

$$L = 861.4, \quad V = 100, \quad VL = 86141 \text{ ft.}^2/\text{sec.};$$

$$\log_{10} VL = 4.93522.$$

From Fig. 9 with  $\log_{10} VL = 4.93522$  pick off  $\log_{10} C_H = 8.147-10$ ;  
 $C_H = .01403$  as explained above.

$$\text{Drag} = .01403 \times \frac{.00237}{2} \times (5,000,000)^{2/3} \times 100^2 = 4860.5 \text{ lb.}$$

Requirement 3:

HP. absorbed in overcoming bare hull drag at 120 ft./sec.

$$L = 861.41 \text{ ft.}; \quad V = 120 \text{ ft./sec.}; \quad VL = 103369 \text{ ft.}^2/\text{sec.},$$

$$\log_{10} VL = 5.01439$$

From Fig. 9 with  $\log_{10} VL = 5.01439$  pick off  
 $\log_{10} C_H = 8.132-10$ ;  $C_H = .01355$

$$\text{Drag} = C_H \frac{\rho}{2} (\text{Vol})^{2/3} V^2 = .01355 \times \frac{.00237}{2} \times (5,000,000)^{2/3} \times 120^2 =$$

$$6761.8 \text{ lb.}$$

$$\text{HP. absorbed} = \frac{\text{Drag } V}{550} = \frac{6761.8 \times 120}{550} = 1475.3 \text{ HP.}$$

Note: HP. to equip ship with =  $\frac{(\text{Hull Drag} + \text{External Drag}) V_{\text{max}}}{550 \times \text{Propeller Efficiency}}$



## Symbols and Formulas

Length	L ft.
Maximum diameter	D ft.
Distance nose to max. dia.	x ft.
Maximum radius	r ft.
(Vol) - air volume	(Vol) cu.ft.
Eccentricity, nose ellipse	e no dimensions
	$e = \frac{\sqrt{x^2 - r^2}}{x}$ no dimensions
Geometric length	$L_g = \sqrt[4]{[(Vol) + \frac{\pi r^3}{3}] L}$ ft.
Cylindrical coef. (Cyl. Coef)	$= \frac{(Vol)}{\frac{\pi D^2 L}{4}} = \frac{4 (Vol)}{\pi D^2 L}$ no dimensions
Fineness ratio	L/D no dimensions
Pressure difference shape coef.	$Y = e (Cyl. Coef.) (L/D)$ no dimensions.
	$Y = e \left( \frac{4 (Vol)}{\pi D^3} \right)$
Skin friction shape coef.	$Z = \frac{L}{L_g} \times \frac{L}{D} = \frac{L^2}{L_g D}$ no dimensions
Whole hull shape coef.	(Y + Z)
Virtual volume	$V_M = (Vol) + \frac{\pi r^3}{3}$ cu.ft.
Density	$\rho$ slugs/cu.ft.
Air speed	V - ft./sec.
VL	Air speed $\times$ length ft. <sup>2</sup> /sec.
Drag	$R = C_H \frac{\rho}{2} (Vol)^{2/3} V^2$ lb.

## Symbols and Formulas (Cont.)

Drag coef. of bare hull  $C_H$  no dimensions

Horsepower absorbed by drag  $R_d$ ;  $HP. = \frac{R V}{550}$

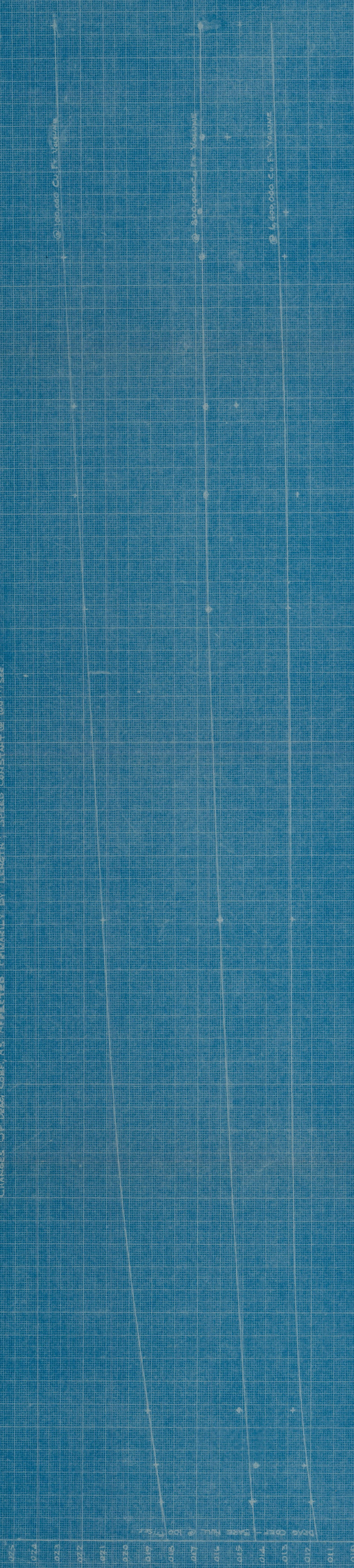
$$\left( \frac{\text{Length at Volume 1}}{\text{Length at Volume 2}} \right)^3 = \frac{\text{Volume 1}}{\text{Volume 2}}$$



PRELIMINARY PLOT No 5

DRAG COEF OF BARE HULL -  $V_e = \frac{L^2}{L^3 D}$  @ 100 F/SEC

CHANGES OF DRAG COEF, AS AFFECTED PRIMARILY BY LENGTH; SPEED CONSTANT @ 100 F/SEC.



← Z →

$$\frac{(\text{LENGTH})^2}{\left[ \sqrt{\left( \text{VOLUME} + \frac{\pi^2}{3} \right) (\text{LENGTH})} \right] (\text{DIAMETER})} = \frac{L^2}{L^3 D}$$

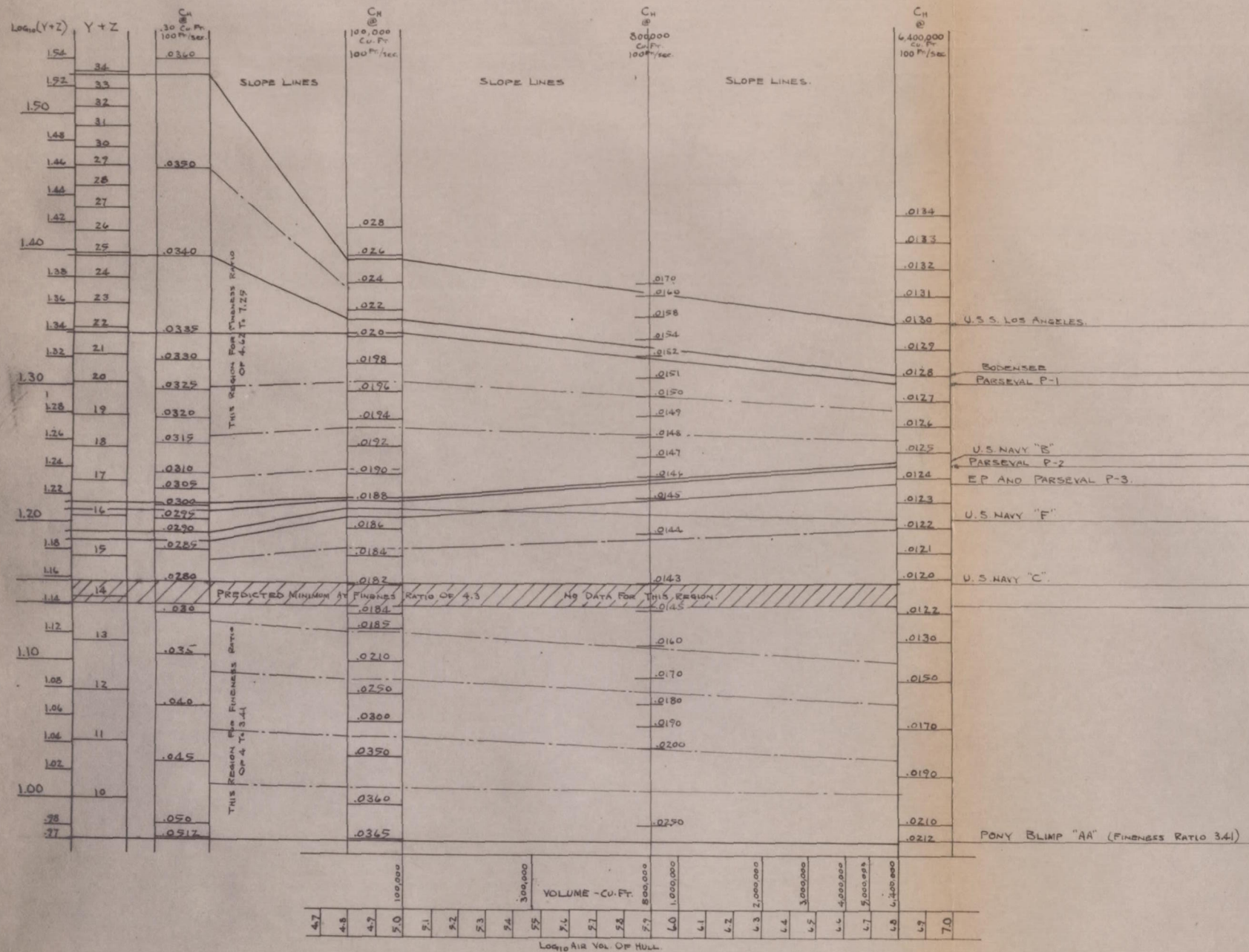
NOTE THIS QUANTITY APPEARS TO AFFECT THAT PART OF THE DRAG COEF DUE TO SKIN FRICTION

CALCULATIONS BY CLINTON H. HAVILL  
LIEUT. U. S. NAVY

FIGURE 5

DIAGRAM SHOWING THE CHANGE OF DRAG COEFFICIENT OF BARE AIRSHIP HULLS  
WITH VOLUME. SPEED CONSTANT @ 100 FT./SEC.  
FROM MODEL TO FULL SIZE - FOR AIRSHIP HULLS WITH CONTINUOUS CURVATURE CONTOUR.

FOR HULLS WITH CONTINUOUS CURVATURE CONTOUR, FINENESS RATIOS 3.41 TO 7.25, CYLINDRICAL COEF. .989 TO .656, ECCENTRICITY OF NOSE ELLIPSE .932 TO .978, AND DISTANCE FROM NOSE TO MAXIMUM ORDINATE 30% TO 47.33% LENGTH.



DEFINITIONS & SYMBOLS FOR THIS DIAGRAM.

$C_H$  = DRAG COEF. OF BARE HULL (NO DIMENSIONS).  
 $C_H = \frac{2K_H}{(\rho U)^2 D^2}$   
 $R_H$  = DRAG OF BARE HULL. (LBS.)  
 $VOL$  = AIR VOL OF HULL. (CU. FT.)  
 $U$  = AIR SPEED. (FT./SEC.)  
 $\rho$  = DENSITY OF AIR ( $\frac{14.7}{14.7 + 32(Y+Z)}$ ) OR ( $\frac{14.7}{32(Y+Z)}$ )

$Y$  = AN EMPIRICAL TERM WHICH PRIMARILY DETERMINES A FUNCTION - THAT PART OF THE DRAG COEF. DUE TO PRESSURE DIFFERENCE.  
 $Y$  = (NO DIMENSIONS)  
 $Y$  = (ECCENTRICITY OF NOSE ELLIPSE)(CYLINDRICAL COEF)(FINENESS RATIO).  
 $Y = (\epsilon) \left( \frac{VOL}{\pi D^2 L} \right) \left( \frac{L}{L_0} \right)$   
 $Y = (\epsilon) \left( \frac{4 VOL}{\pi D^2} \right)$

WHERE

$D$  = MAX DIAM. (FEET)  
 $VOL$  = VOLUME (CU. FT.)  
 $\epsilon$  = ECCENTRICITY OF NOSE ELLIPSE.  
 $L = \frac{\sqrt{2^2 - \epsilon^2}}{\epsilon}$

WHERE

$X$  = DISTANCE ALONG AXIS OF SHIP, FROM NOSE TO FIRST POINT OF MAX DIAM.  
 $Y$  = MAX RADIUS =  $\frac{DIAM}{2}$   
 CYLINDRICAL COEF. =  $\frac{VOLUME}{\pi D^2 L}$

FINENESS RATIO =  $\frac{L}{D}$

$Z$  = AN EMPIRICAL TERM WHICH PRIMARILY DETERMINES A FUNCTION - THAT PART OF THE DRAG COEF. DUE TO SKIN FRICTION.  
 $Z$  = (NO DIMENSIONS)

$Z = \left( \frac{LENGTH}{GEOMETRIC LENGTH} \right) (FINENESS RATIO)$   
 $Z = \left( \frac{L}{L_0} \right) \left( \frac{L}{D} \right) = \frac{L^2}{L_0 D}$

WHERE

$L$  = LENGTH OF HULL (FT.)  
 $L_0$  = A TERM OF LINEAR DIMENSIONS USED TO COMPARE SHIPS AT THE SAME  $U_L$  - DEFINED IN TEXT AS "GEOMETRIC LENGTH".  
 $L_0 = \sqrt[3]{(VIRTUAL VOLUME)(LENGTH)}$   
 $L_0 = \sqrt[3]{VOL}$   
 WHERE  $V_0 = (VOL \text{ OF SHIP} + \frac{\pi D^3}{6})$  (CU. FT.)  
 $L_0 = \frac{L \cdot LENGTH}{L_0} (F = F \text{ (SEE ABOVE)})$   
 $L_0 = (FEET)$

THIS DIAGRAM FOR STANDARD VALUE OF  $\frac{\rho}{\rho_0}$

DIRECTIONS FOR USE -

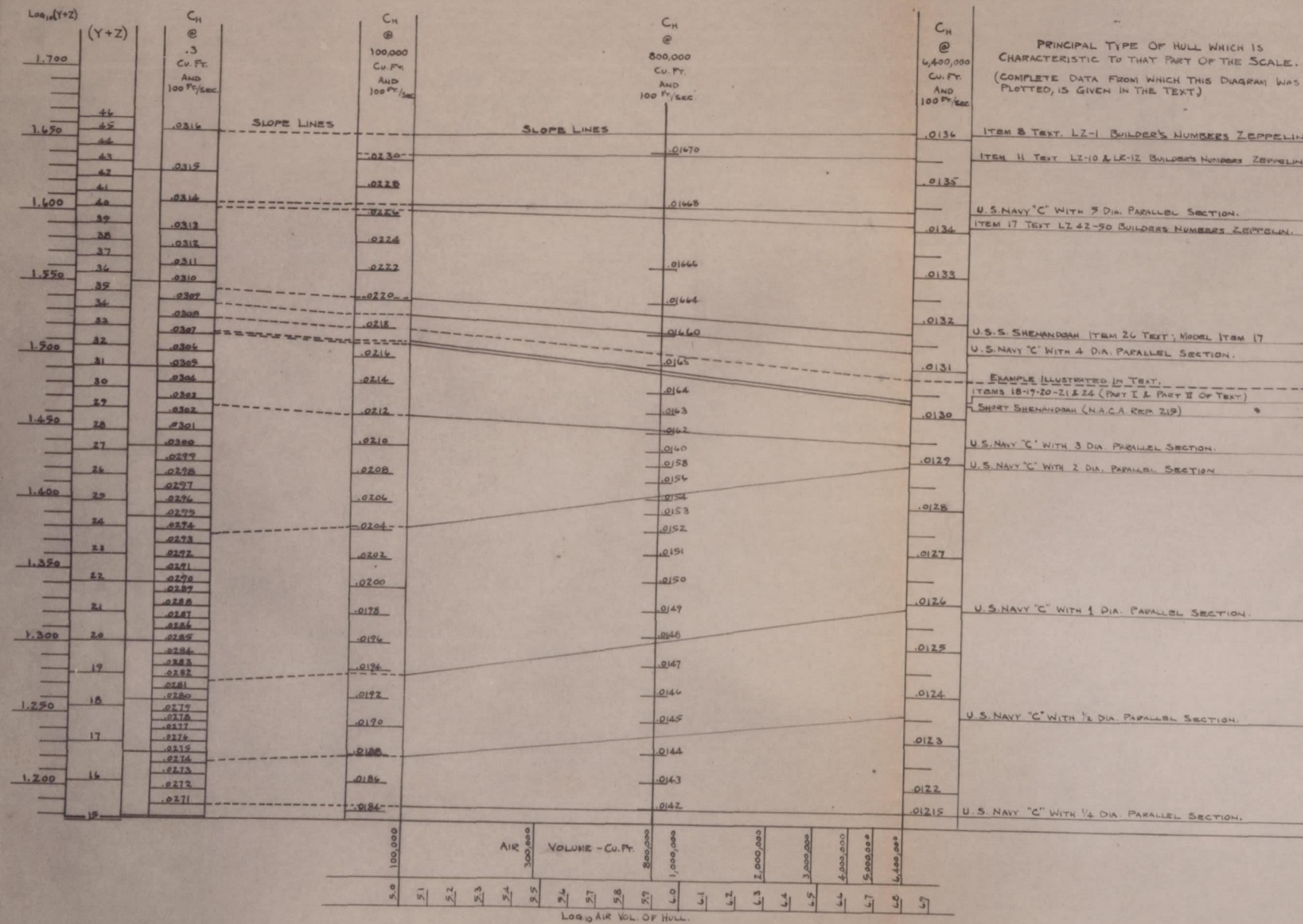
CALCULATE 'Y' & 'Z' AND ADD THEM. ENTER THE LEFT HAND SCALE WITH THE VALUE OF  $\log_{10}(Y+Z)$  AND FOLLOW THROUGH HORIZONTALLY TO SCALE OF  $C_H$  @ .30 CU. FT. THEN FOLLOW ACROSS THE DIAGRAM, INTERPOLATING THE SLOPE BETWEEN THE ADJACENT SLOPE LINES. PICK OFF THE DRAG COEF'S AT VOLUMES OF 100,000 - 800,000 & 6,400,000 CU. FT. THESE DRAG COEF'S, ARE FOR THE VOLUMES GIVEN AT 100 FT./SEC AIR SPEED. FOR INTERPOLATION FOR VOLUMES OTHER THAN THOSE GIVEN, PROCEED AS FOLLOWS. WITH THE THREE VALUES OF DRAG COEF, AS PICKED OFF FROM THE DIAGRAM, CALLED  $C_H-100000$  -  $C_H-800000$  &  $C_H-6400000$ , PLOT THE  $\log_{10}$  OF THESE THREE VALUES AGAINST THE  $\log_{10}$  OF  $(LENGTH \times 100)$  WHERE LENGTH IS THE LENGTH AT VOLUMES OF 100,000 - 800,000 & 6,400,000 CU. FT. FOR REDUCTION OF EXPANSION  $\left[ \frac{ACTUAL LENGTH OF PARTICULAR SHIP}{LENGTH AT VOLUME "X"} \right]^3 = \frac{VOLUME OF PARTICULAR SHIP}{VOLUME "X"}$ . THE CURVE OF  $\log_{10}$  DRAG COEF VS  $\log_{10} DL$  DIFFERS WITH EACH TYPE OF SHIP. PASS A SMOOTH CURVE THROUGH THE THREE POINTS ESTABLISHED (USE REASONABLY SMALL DL SCALE) AND PICK OFF THE VALUE OF DRAG COEF, THAT CORRESPONDS TO A DL OF 100 TIMES LENGTH OF THE PARTICULAR SHIP. FOR FURTHER INFORMATION SEE PART II OF TEXT. SEE EXAMPLE FOR ILLUSTRATING USE OF THIS DIAGRAM IN PART II OF TEXT.

CALCULATIONS BY *Clinton H. Havill*  
LIEUT. U. S. NAVY.

DIAGRAM SHOWING THE CHANGE OF DRAG COEFFICIENT OF BARE AIRSHIP HULLS  
WITH VOLUME, SPEED CONSTANT @ 100 FT./SEC.

FROM MODEL TO FULL SIZE - FOR AIRSHIP HULLS WITH PARALLEL SECTIONS.

LIMITS OF THE DATA FROM WHICH THIS WAS DERIVED - FOR HULLS WITH PARALLEL SECTION EQUAL TO NOT LESS THAN 1/4 OF THE DIAMETER, FINENESS RATIO FROM 4.85 TO 10.61, CYLINDRICAL COEFFICIENT FROM .674 TO .914, ECCENTRICITY OF NOSE ELLIPSE FROM .932 TO .984, AND DISTANCE FROM NOSE TO FIRST MAXIMUM ORDINATE FROM 14.37% TO 30.80% OF THE LENGTH.



DEFINITIONS & SYMBOLS FOR THIS DIAGRAM.

$C_H$  = DRAG COEF. OF BARE HULL (NO DIMENSIONS).  
 $C_H = \frac{2R}{\rho(VOL)^{2/3} D^2}$   
 $R_H$  = DRAG OF BARE HULL (LBS.)  
 $VOL$  = AIR VOL. OF HULL. (Cu. Ft.)  
 $U$  = AIR SPEED. (FT./SEC.)  
 $\rho$  = DENSITY OF AIR ( $\frac{1}{35}$  LBS./Cu. Ft.) OR ( $\frac{1}{1.225}$  SLUGS/Cu. Ft.)

$Y$  = AN EMPIRICAL TERM WHICH PRIMARILY DETERMINES A FUNCTION - THAT PART OF THE DRAG COEF. DUE TO PRESSURE DIFFERENCE.  
 $Y$  = (NO DIMENSIONS)  
 $Y$  = (ECCENTRICITY OF NOSE ELLIPSE)(CYLINDRICAL COEF.)(FINENESS RATIO).  
 $Y = (\lambda) \left( \frac{VOL}{D^3} \right) \left( \frac{L}{D} \right)$   
 $Y = (\lambda) \left( \frac{4 VOL}{\pi D^3} \right)$

WHERE  
 $D$  = MAX. DIAM. (FEET.)  
 $VOL$  = VOLUME (Cu. Ft.)  
 $\lambda$  = ECCENTRICITY OF NOSE ELLIPSE.  
 $\lambda = \frac{\sqrt{X^2 - Y^2}}{X}$

WHERE  
 $X$  = DISTANCE, ALONG AXIS OF SHIP, FROM NOSE TO FIRST POINT OF MAX. DIAM.  
 $Y$  = MAX. RADIUS =  $\frac{D_{MAX}}{2}$   
 CYLINDRICAL COEF. =  $\frac{VOLUME}{\pi D^2 L}$   
 FINENESS RATIO =  $\frac{L}{D}$

$Z$  = AN EMPIRICAL TERM WHICH PRIMARILY DETERMINES A FUNCTION - THAT PART OF THE DRAG COEF. DUE TO SKIN FRICTION.  
 $Z$  = (NO DIMENSIONS)  
 $Z = \left( \frac{LENGTH}{GEOMETRIC LENGTH} \right) (FINENESS RATIO)$   
 $Z = \left( \frac{L}{L_g} \right) \left( \frac{L}{D} \right) = \frac{L^2}{L_g D}$

WHERE  
 $L$  = LENGTH OF HULL (FT.)  
 $L_g$  = A TERM OF LINEAR DIMENSIONS USED TO COMPARE SHIPS AT THE SAME D.L. - DEFINED IN TEXT AS "GEOMETRIC LENGTH".  
 $L_g = \frac{\sqrt{VOLUME} (LENGTH)}{\sqrt{VOLUME}}$   
 $L_g = \sqrt{VOLUME}$   
 WHERE  $VOLUME = (VOL \text{ OF SHIP} + \frac{\pi D^3}{6})$  (Cu. Ft.)  
 $L_g = \frac{\sqrt{VOL + \frac{\pi D^3}{6}} (L)}{L} \quad [r = r_1 \text{ (SEE ABOVE)}]$   
 $L_g = (\text{FEET.})$

THIS DIAGRAM FOR STANDARD VALUE OF  $\frac{C_H}{V}$ .

DIRECTIONS FOR USE :-

CALCULATE "Y" & "Z" AND ADD THEM. ENTER THE LEFT HAND SCALE WITH THE VALUE OF  $Log_{10}(Y+Z)$  AND FOLLOW THROUGH HORIZONTALLY TO SCALE OF  $C_H @ .30$  Cu. Ft. THEN FOLLOW ACROSS THE DIAGRAM, INTERPOLATING THE SLOPE BETWEEN THE ADJACENT SLOPE LINES. PICK OFF THE DRAG COEF.'S AT VOLUMES OF 100,000 - 800,000 & 6,400,000 Cu. Ft. THESE DRAG COEF.'S ARE FOR THE VOLUMES GIVEN, PROCEED AS FOLLOWS. WITH THE THREE VALUES OF DRAG COEF. AS PICKED OFF FROM THE DIAGRAM, CALLED  $C_{H100000}$ ,  $C_{H800000}$  &  $C_{H6400000}$ , PLOT THE  $Log_{10}$  OF THESE THREE VALUES AGAINST THE  $Log_{10}$  OF (LENGTH x 100) WHERE LENGTH IS THE LENGTH AT VOLUMES OF 100,000 - 800,000 & 6,400,000 Cu. Ft. FOR REDUCTION OR EXPANSION [ACTUAL LENGTH OF PARTICULAR SHIP]  $\times$  [VOLUME OF PARTICULAR SHIP]  $\div$  [VOLUME OF 100,000 Cu. Ft.]. THE CURVE OF  $Log_{10}$  DRAG COEF. VS  $Log_{10}$  DL DIFFERS WITH EACH TYPE OF SHIP. PASS A SMOOTH CURVE THROUGH THE THREE POINTS ESTABLISHED (USE REASONABLY SMALL DL SCALE) AND PICK OFF THE VALUE OF DRAG COEF. THAT CORRESPONDS TO A DL OF 100 TIMES LENGTH OF THE PARTICULAR SHIP. FOR FURTHER INFORMATION SEE PART II OF TEXT. SEE EXAMPLE FOR ILLUSTRATING USE OF THIS DIAGRAM IN PART II OF TEXT.

CALCULATIONS BY *Clinton H. Hawill*  
LIEUT. U.S. NAVY.

FIGURE 8.

Plot of Wind Tunnel Results on the Seventeen Models Tabulated in the Text.

Drag of Bare Hull (Model) vs  $\log_{10} U L^2 g$

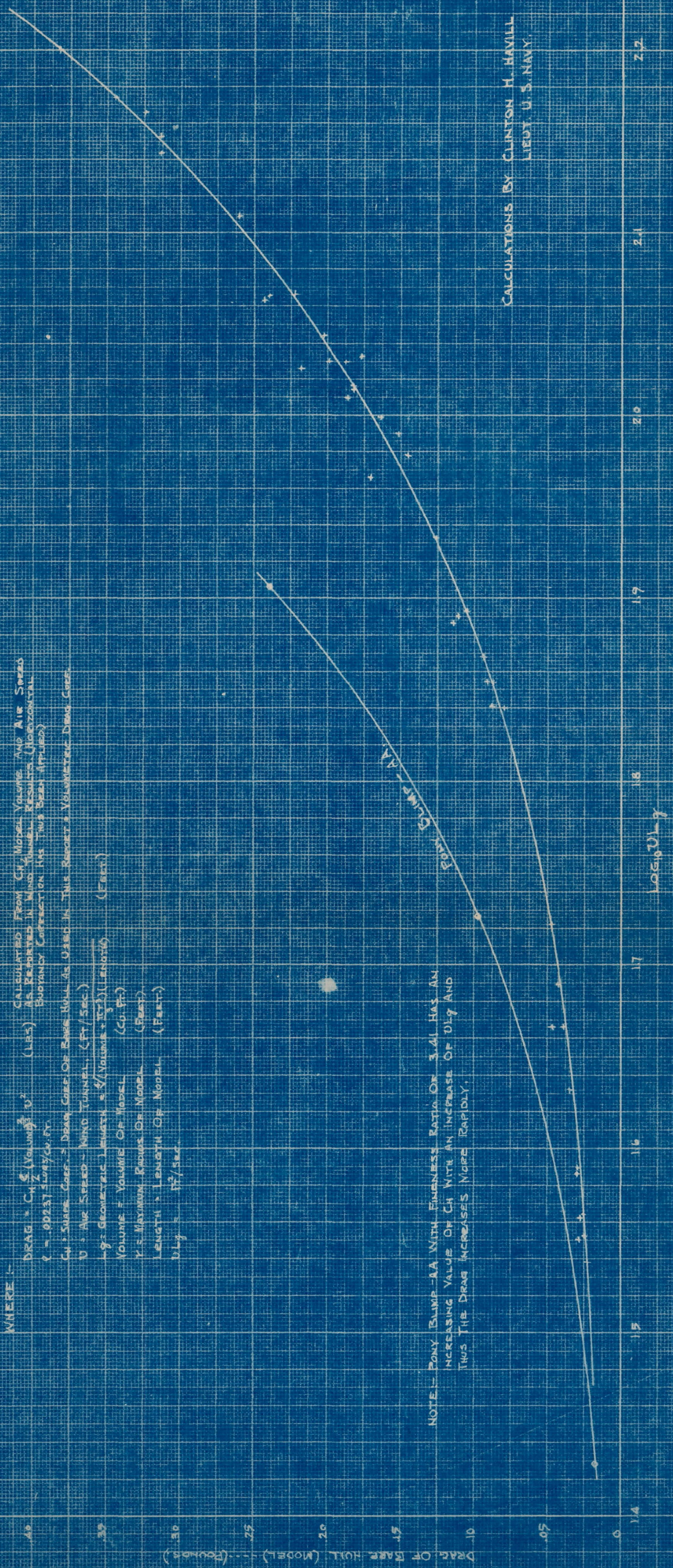
WHERE -

$DRAG = C_D \frac{\rho}{2} (Volume)^2 U^2$  (LBS)  
 $\rho = .00237 \text{ Slugs/Cu. Ft.}$   
 $C_D = \text{Drag Coef.} = \text{Drag Coef. of Bare Hull as Used in This Report's Volumetric Drag Coef.}$   
 $U = \text{Air Speed - Wind Tunnel (Ft./Sec.)}$   
 $g = \text{Geometric Length} = \sqrt{\text{Volume} \times \pi} / (\text{Length})$  (Feet)  
 $(Volume = \text{Volume of Model (Cu. Ft.)})$   
 $\pi = \text{Maximum Radius of Model (Feet)}$   
 $\text{LENGTH} = \text{Length of Model (Feet)}$   
 $U L^2 g = \text{Ft}^2 / \text{Sec.}$

CALCULATED FROM  $C_D$  Model Volume and Air Speed  
 AS REPORTED IN WIND TUNNEL RESULTS. (Horizontal  
 Buoyancy Correction Has Thus Been Applied)

NOTE - PONY BUMP - AA WITH FINNESS RATIO OF 3.41 HAS AN  
 INCREASING VALUE OF  $C_D$  WITH AN INCREASE OF  $U L^2 g$  AND  
 THUS THE DRAG INCREASES MORE RAPIDLY.

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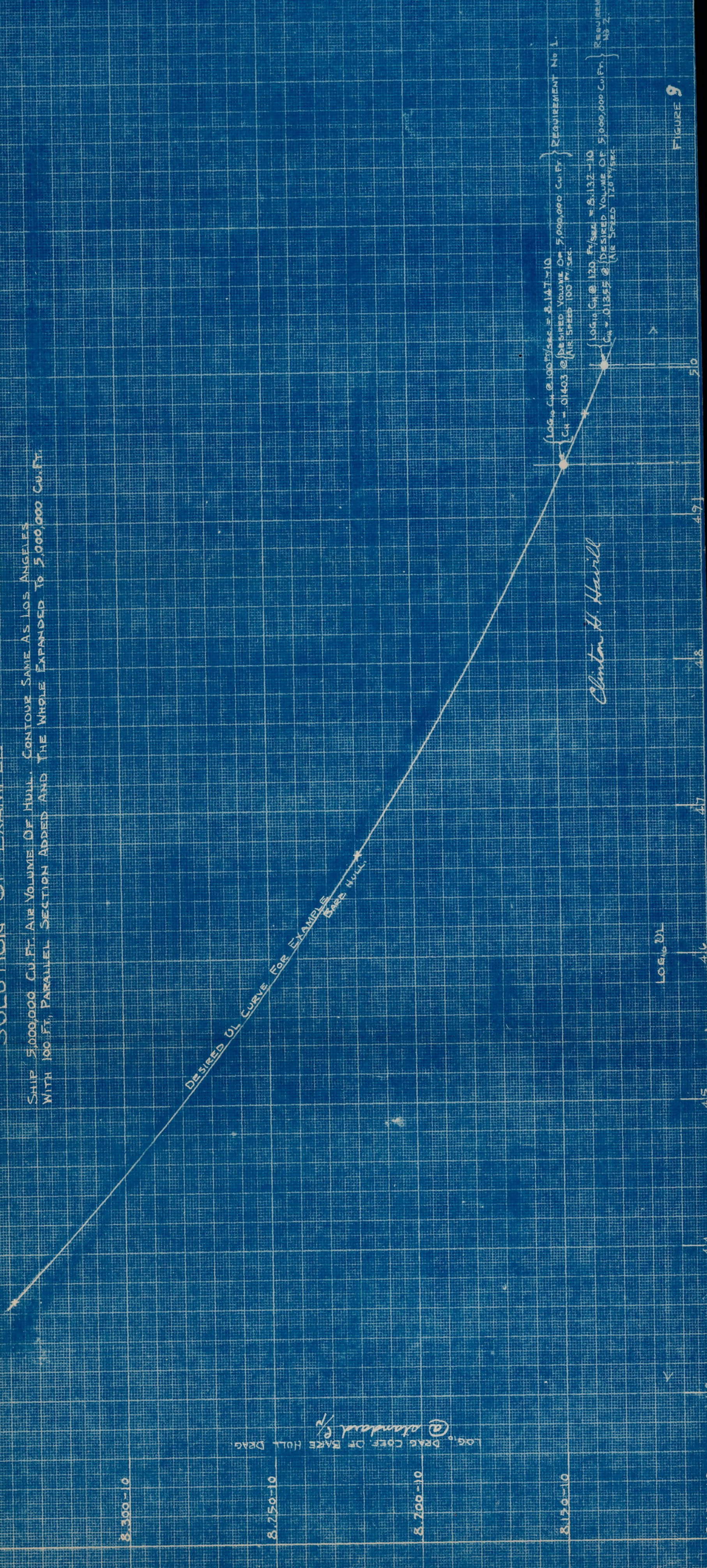


**PART II**  
**SOLUTION OF EXAMPLE**

SHIP 5,000,000 CU.FT. AIR VOLUME OF HULL. CONTOUR SAME AS LOS ANGELES  
 WITH 100 FT. PARALLEL SECTION ADDED AND THE WHOLE EXPANDED TO 5,000,000 CU.FT.

DESIRED U<sub>L</sub> CURVE FOR EXAMPLES  
 BARE HULL

LOG<sub>10</sub> DRAG COEF. OF BARE HULL DRAG  
 @ standard 1/2"



Clinton H. Havill

FIGURE 9



PART II

PRELIMINARY PLOT NO 4

DRAG COEFF. OF BARE HULL - VS - (ECCENTRICITY OF NOSE ELLIPSE) (PRISMATIC COEFF) (FINENESS RATIO)

THIS PLOT TO INDICATE RELATION BETWEEN DRAG COEFF, DIAMETER EFFECT, WITH VOLUME, WITH VELOCITY CONSTANT @ 100 FT/SEC.

@ 100,000 Cu. Ft. Volume

@ 800,000 Cu. Ft. Volume  
 @ 6,400,000 Cu. Ft. Volume

0.25  
0.24  
0.23  
0.22  
0.21  
0.20  
0.19  
0.18  
0.17  
0.16  
0.15  
0.14  
0.13  
0.12  
0.11  
0.10  
0.09

DRAG COEFF. - BARE HULL @ 100 FT/SEC



(ECCENTRICITY OF NOSE ELLIPSE) (PRISMATIC COEFF) (FINENESS RATIO) =  $(\frac{VOL}{\pi D^3}) \cdot (\frac{VOL}{L^3}) = \frac{VOL}{\pi D^3 L^3}$

- ← Note this quantity is independent of length except as it appears volume.
- ← Note this quantity has the maximum effect of diameter, as diameter enters as the third power.
- ← Note this quantity appears to appear that part of the drag coeff due to pressure difference.

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FIGURE 4

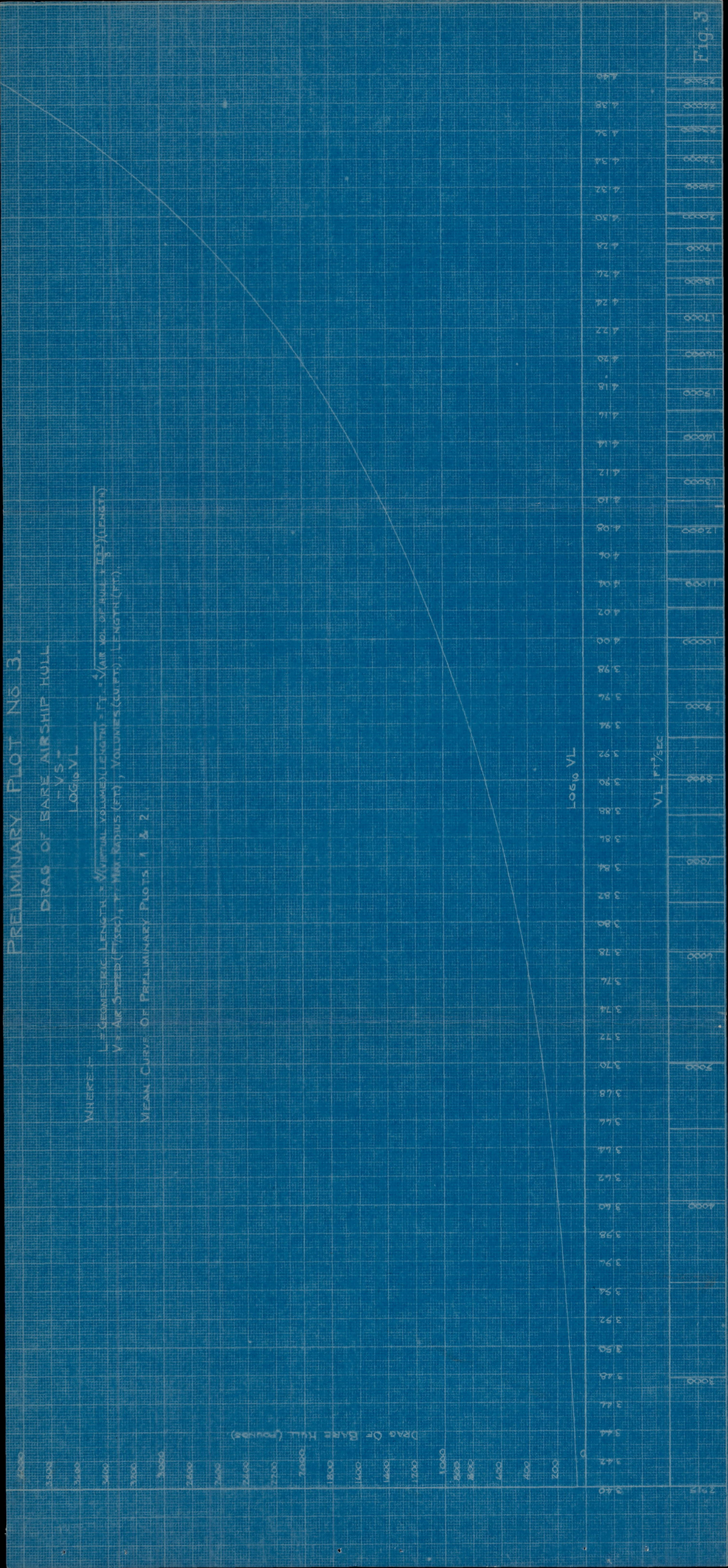
# PRELIMINARY PLOT NO 3.

## DRAG OF BARE AIRSHIP HULL

-VS-  
LOG<sub>10</sub> VL

WHERE:  $L = \sqrt[3]{\text{VIRTUAL VOLUME}(\text{LENGTH})} = \sqrt[3]{\frac{V}{\rho}}$  (AIR VOL OF HULL +  $\frac{\pi R^2 L}{2}$ ) (LENGTH)  
 $V = \text{AIR SPEED}(\text{FT/SEC})$ ,  $R = \text{MAX RADIUS}(\text{FT})$ ,  $\text{VOLUMES}(\text{CU.FT})$ ,  $\text{LENGTH}(\text{FT})$ .

MEAN CURVE OF PRELIMINARY PLOTS 1 & 2.



# PRELIMINARY PLOT NO 2

DRAG OF BARE AIRSHIP HULL  
VS  
LOG<sub>10</sub> VL

WHERE -  
 $L$  = GEOMETRIC LENGTH =  $\sqrt[3]{(\text{AIR VOL. OF HULL} \cdot \pi \cdot r^2) \cdot (\text{LENGTH})}$   
 $V$  = AIR SPEED (F/SEC);  $r$  = MAX RADIUS (FT); VOLUMES (CUBIC FT), LENGTH (FT).

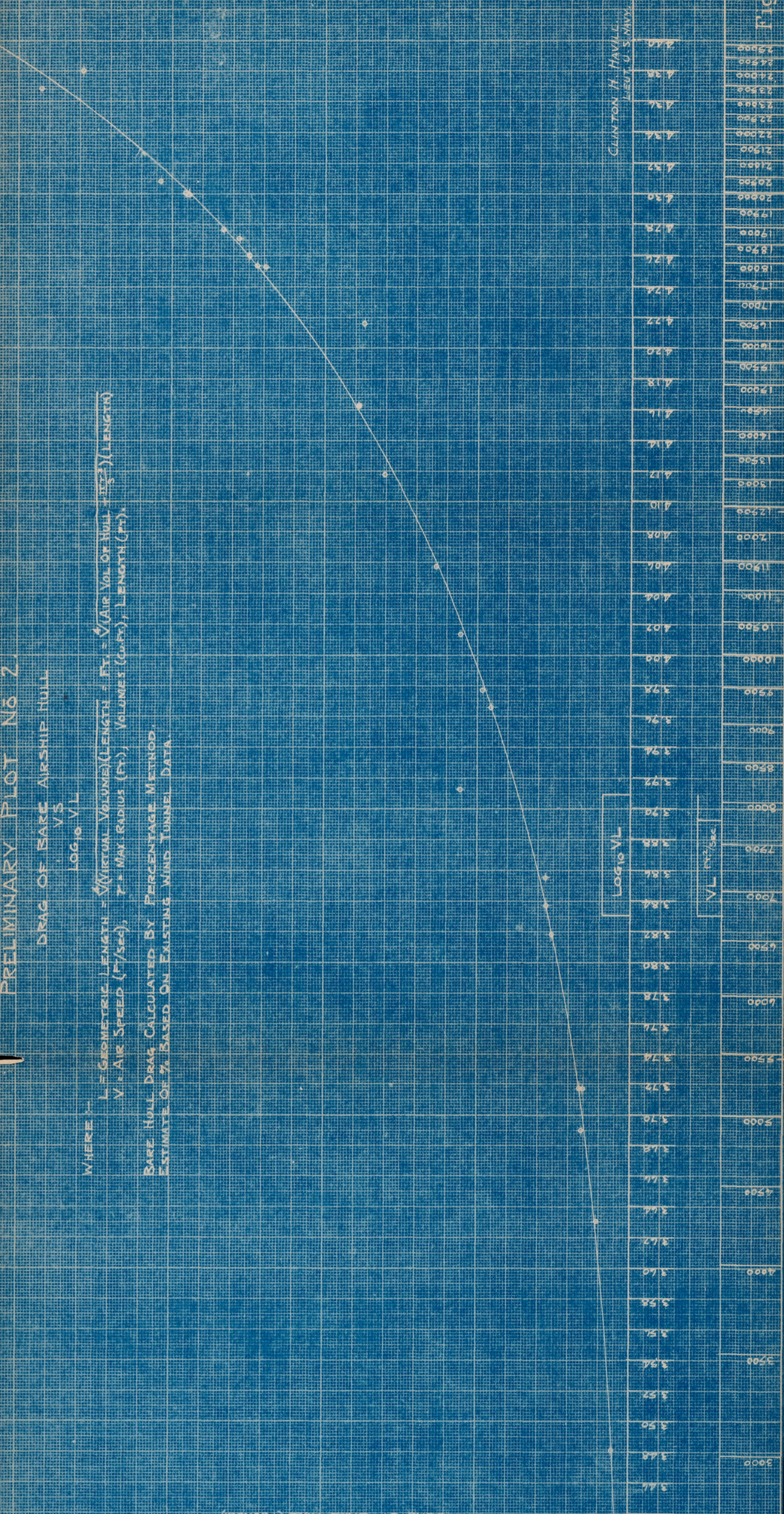
BARE HULL DRAG CALCULATED BY PERCENTAGE METHOD.  
 ESTIMATE OF % BASED ON EXISTING WIND TUNNEL DATA.

DRAG OF BARE HULL (POUNDS)

LOG<sub>10</sub> VL

VL (FT/SEC)

4000  
3800  
3600  
3400  
3200  
3000  
2800  
2600  
2400  
2200  
2000  
1800  
1600  
1400  
1200  
1000  
800  
600  
400  
200  
0  
3.40  
3.42  
3.44  
3.46  
3.48  
3.50  
3.52  
3.54  
3.56  
3.58  
3.60  
3.62  
3.64  
3.66  
3.68  
3.70  
3.72  
3.74  
3.76  
3.78  
3.80  
3.82  
3.84  
3.86  
3.88  
3.90  
3.92  
3.94  
3.96  
3.98  
4.00  
4.02  
4.04  
4.06  
4.08  
4.10  
4.12  
4.14  
4.16  
4.18  
4.20  
4.22  
4.24  
4.26  
4.28  
4.30  
4.32  
4.34  
4.36  
4.38  
4.40  
4.42  
4.44  
4.46  
4.48  
4.50  
4.52  
4.54  
4.56  
4.58  
4.60  
4.62  
4.64  
4.66  
4.68  
4.70  
4.72  
4.74  
4.76  
4.78  
4.80  
4.82  
4.84  
4.86  
4.88  
4.90  
4.92  
4.94  
4.96  
4.98  
5.00



CLINTON H. HAVILL  
 LEUT. U. S. NAVY

Drag of Bare Airship Hull  
vs  
 $\log_{10} VL$

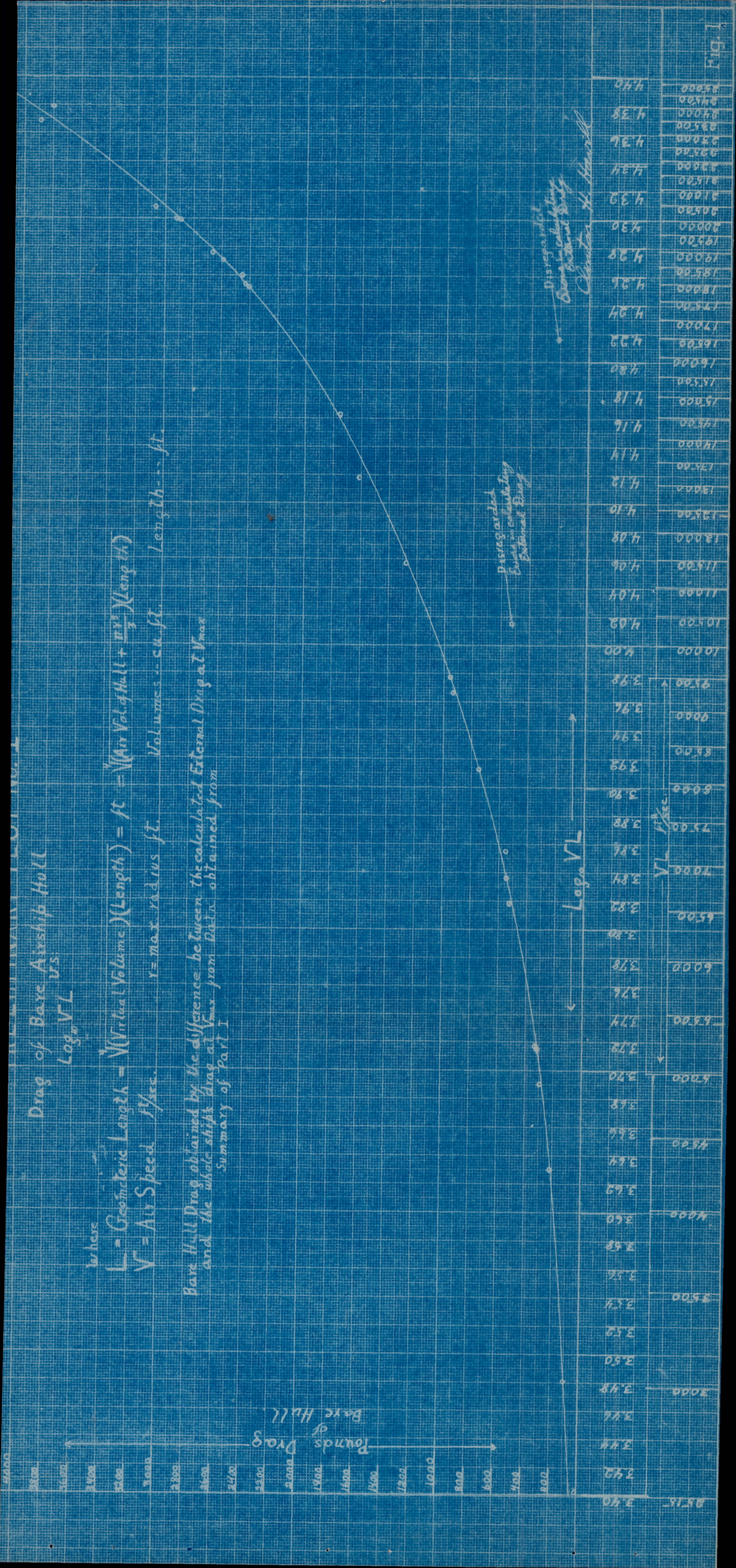
where  
 $L = \text{Geometric Length} = \sqrt[3]{\text{Virtual Volume}} \times (\text{Length}) = ft = \sqrt[3]{(\text{Air Vol. of Hull} + \frac{\pi r^3}{3})} \times (\text{Length})$   
 $V = \text{Air Speed} \text{ M/sec.}$   
 $r = \text{max radius ft.}$   
 $\text{Volumes} = \text{cu ft.}$   
 $\text{Length} = \text{ft.}$

Bare Hull Drag obtained by the difference between the calculated External Drag at  $V_{max}$  and the wheel strips drag at  $V_{max}$  from data obtained from Summary of Part I

Pounds of Drag of Bare Hull

$\log_{10} VL$

$VL \text{ ft/sec}$



Disregarded Points in calculating External Drag  
Character of Hull

Disregarded Points in calculating External Drag

FINAL SUMMARY OF PART II  
ARRANGEMENT OF PREVIOUS DATA IN ASCENDING VALUES OF "Y + Z"

SHIPS	"Y" + "Z"	DRAG COEFFICIENT - C <sub>H</sub>				FIGURE 8	
		@ Cu. Ft. .3	@ 100,000 Cu. Ft.	@ 800,000 Cu. Ft.	@ 6,400,000 Cu. Ft.		
"AA"	9.377	.0512	.03620	.02600	.02120	CONTINUOUS CURVATURE	
"C"	14.356	.0280	.01824	.01430	.01193		
"EP"	15.396	.0285	.01868	.01442	.01228		
"F"	15.588	.0290	.01878	.01445	.01233		
"P-3"	16.165	.0295	.01880	.01458	.01236		
"P-2"	16.177	.0298	.01881	.01458	.01245		
"B"	16.468	.0301	.01882	.01459	.01247		
"P-1"	21.811	.0335	.02019	.01519	.01278		
BODENSEE	24.904		.02070	.01522	.01280		
LOS ANGELES	33.696		.02480	.01610	.01300		
"C" + 1/4 DIA.	15.413	.0271	.01842	.01442	.01215		PARALLEL SECTION
"C" + 1/2 DIA.	16.548	.02741	.01880	.01446	.01236		
"C" + 1 DIA.	18.779	.02819	.01939	.01478	.01259		
"C" + 2 DIA.	23.807	.02938	.02042	.01542	.01288		
"C" + 3 DIA.	28.998	.03026	.02120	.01622	.01294		
SHORT SHENANDOAH	32.744	.03068	.02164	.01647	.01303		
LZ-72 TO 90 EXCEPT 73, 77 & 81.	32.898		.02164	.01648	.01304		
LZ-91 TO 94, 95 TO 99, 100, 101 & 106 TO 111	32.907		.02165	.01649	.01305		
"C" + 4 DIA. SHENANDOAH.	34.346	.03088	.02190	.01661	.01340		
LZ-42 TO 50.	35.122	.03090	.02201	.01666	.01347		
LZ-102 & 104.	35.273		.02238	.01666	.01348		
LZ-59 TO 61, 64 TO 71 EXCEPT 60 & 70.	37.372		.02258	.01667	.01349		
"C" + 5 DIA.	39.982		.02262	.01668	.01342		
LZ-112 TO 114.	40.358	.0314	.02263	.01668	.01344		
LZ-10 & 12.	40.650		.02270	.01670	.01350		
LZ-1.	43.205		.02310	.01680	.01356		
LZ-4 & 5.	45.229		.02320	.01683	.01362		
LZ-7 & 8.	46.398		.02420	.01699	.01392		
	47.085		.02490	.01720	.01400		

NOTES.

THE SCALES ON FIGURES 7 & 8 WERE PLOTTED WITH LOG<sub>10</sub>(Y+Z) A UNIFORM SCALE OF LOGARITHMS. THE MODEL SCALE WAS CONSTRUCTED NEARLY UNIFORM AND THE SCALE @ 100,000 WAS CALIBRATED ON THIS DATA. THE SCALE @ 6,400,000 WAS CONSTRUCTED NEARLY UNIFORM AND THE SLOPE LINES DRAWN IN FROM SCALE @ 100,000. THE SCALE @ 800,000 WAS ALLOWED TO CALIBRATE ITSELF ON THE DATA GIVEN HERE. CURVES OF THE SCALES WERE DRAWN AND THE GRADUATIONS MARKED WERE THUS TRANSFERRED BACK TO THE SCALE.

IT IS THUS SEEN THAT THE SCALES ARE EMPIRICALLY CALIBRATED ON THE DATA HERE, MAKING THE SLOPE LINES STRAIGHT LINES AND GRADUATING THE SCALES ACCORDINGLY.

CALCULATIONS BY CLINTON H. HAVILL  
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PART II

ITEM	A SHAPE COMPARISON OF HULL DRAG COEF @ 100 FT/SEC										PRELIMINARY PLOT NO. 5.					
	Ecc. CYL. $\frac{L}{D}$ *	WHEN THE SHIPS CONNECTED BY ARROWS ARE REDUCED TO THE SAME VOLUME THEY ARE IDENTICAL TO THE MEAN OF THEIR DRAG COEFS CAN BE TAKEN FOR ANY OF THEM.	MEAN EXC. $\frac{L}{D}$ FOR ITEMS CONNECTED "Y"	MEAN VALUE OF DRAG COEF OF HULL			LOG Lg	LOG LENGTH	LOG $\frac{L}{Lg}$ (LENGTH $\sqrt[4]{\frac{L}{Lg}}$ )	FINENESS RATIO $\frac{L}{D}$	$\frac{L^2}{LgD}$	MEAN VALUE OF DRAG COEF. OF HULL.			MEAN VALUE OF $\frac{L}{LgD}$ OF SHIPS CONNECTED.	
				@ 100,000 Cu. Ft. Vol.	@ 800,000 Cu. Ft. Vol.	@ 6,400,000 Cu. Ft. Vol.						@ 100,000 Cu. Ft. Vol.	@ 800,000 Cu. Ft. Vol.	@ 6,400,000 Cu. Ft. Vol.		
1	4.21		4.21	.01823	.01422	.01322	1.79207	2.21219	.42012	2.631	5.06	13.31	.01919	.01518	.01280	13.31
2	3.86		3.63	.01705	.01293	.01210	1.89259	2.29226	.39967	2.510	4.62	11.59	.01851	.01439	.01195	11.71
3	3.40						1.89927	2.29667	.39740	2.509	4.72	11.84				
4	3.10		3.10	.01740	.01329	.01240	1.80218	2.20952	.40734	2.555	4.84	12.37	.01880	.01469	.01229	12.37
5	3.10						1.80218	2.20952	.40734	2.555	4.84	12.37				
6	4.04		4.32	.01992	.01378	.01297	2.13711	2.63043	.49332	3.113	6.70	20.86	.02115	.01600	.01284	21.89
7	4.61						2.31877	2.81842	.49965	3.160	7.25	22.91				
8	7.20		7.27	.02232	.01635	.01424	2.06034	2.63144	.57110	3.724	10.21	38.02	.02319	.01676	.01501	37.56
9	9.21		9.39	.02341	.01641	.01562	2.10360	2.64933	.54573	3.513	10.50	36.89				
10	9.76						2.13994	2.68664	.54670	3.522	10.60	37.33				
11	8.14						2.11786	2.66276	.54490	3.507	10.00	35.07	.02317	.01670	.01559	35.59
12	8.12		8.24	.02314	.01623	.01571	2.13719	2.66839	.53120	3.398	9.55	32.45				
13	8.45						2.15339	2.70927	.55588	3.596	10.48	37.69				
14	9.21						2.16407	2.71517	.55108	3.557	10.61	37.74				
15	7.55						2.16999	2.72428	.55429	3.583	10.08	36.12				
16	6.36						2.20659	2.72916	.52257	3.331	8.68	28.91	.02233	.01660	.01526	30.92
17	7.05						2.22803	2.76790	.53987	3.466	9.50	32.93				
18	5.55		5.53	.01964	.01631	.01314	2.28857	2.80956	.52099	3.319	8.24	27.35	.02174	.01650	.01300	27.37
19	5.92						2.28813	2.80956	.52143	3.322	8.24	27.37				
20	5.57						2.28813	2.80956	.52143	3.322	8.24	27.37				
21	5.53		5.89	.02019	.01654	.01316	2.28813	2.80956	.52143	3.322	8.24	27.37	.02279	.01670	.01310	33.48
22	5.89						2.32599	2.87216	.54617	3.517	9.52	33.48				
23	5.89						2.32599	2.87216	.54617	3.517	9.52	33.48				
24	5.53		6.37	.02012	.01662	.01324	2.28813	2.80956	.52143	3.322	8.24	27.37	.02272	.01680	.01300	34.27
25	6.38						2.31989	2.87216	.55627	3.600	9.52	34.27				
26	5.84						2.30105	2.83264	.53199	3.401	8.64	29.38	.02233	.01660	.01259	29.38
"Y"																

\* Ecc. x CYL. x  $\frac{L}{D}$  = (ECCENTRICITY OF NOSE ELLIPSE) x (CYLINDRICAL COEFFICIENT) x (FINENESS RATIO).

CALCULATIONS BY CLINTON H. HAVILL  
LIEUT. U. S. NAVY.



COMPARISON OF BARE HULLS AT VARYING AIR VOLUMES AND A VELOCITY OF 100 FEET PER SECOND.

ITEM	AIR VOLUME OF HULL FROM PART I CU. FT.	100,000 Cu. Ft.			200,000 Cu. Ft.			400,000 Cu. Ft.			800,000 Cu. Ft.			1,600,000 Cu. Ft.			6,400,000 Cu. Ft.			LOG <sub>10</sub> OF ACTUAL VELOCITY AT 100 FT/SEC (AS BEFORE)	DH FROM PART 3 BARE HULL DRAG @ 100 FT/SEC (AS BEFORE) (LBS)	C <sub>H</sub> DRAG COEF OF BARE HULL @ 100 FT/SEC		
		LOG <sub>10</sub> Lg WHEN AIR VOL = 100,000 CU. FT. (LOG Lg + 1.66667)	LOG <sub>10</sub> Lg	LOG <sub>10</sub> Lg + 1.66667	LOG <sub>10</sub> U <sub>10</sub> @ 100 FT/SEC AND 200,000 CU. FT. VOL. (LOG Lg + 1.0034)	LOG <sub>10</sub> U <sub>10</sub> @ 100 FT/SEC AND 400,000 CU. FT. VOL. (LOG Lg + 1.0034)	LOG <sub>10</sub> U <sub>10</sub> @ 100 FT/SEC AND 800,000 CU. FT. VOL. (LOG Lg + 1.0034)	LOG <sub>10</sub> U <sub>10</sub> @ 100 FT/SEC AND 1,600,000 CU. FT. VOL. (LOG Lg + 1.0034)	LOG <sub>10</sub> U <sub>10</sub> @ 100 FT/SEC AND 6,400,000 CU. FT. VOL. (LOG Lg + 1.0034)	C <sub>H</sub> DRAG COEF OF BARE HULL DRAG @ 100 FT/SEC (AS BEFORE)	DH FROM PART 3 BARE HULL DRAG @ 100 FT/SEC (AS BEFORE) (LBS)	C <sub>H</sub> DRAG COEF OF BARE HULL @ 100 FT/SEC (AS BEFORE)	DH FROM PART 3 BARE HULL DRAG @ 100 FT/SEC (AS BEFORE) (LBS)	C <sub>H</sub> DRAG COEF OF BARE HULL @ 100 FT/SEC (AS BEFORE)	DH FROM PART 3 BARE HULL DRAG @ 100 FT/SEC (AS BEFORE) (LBS)	C <sub>H</sub> DRAG COEF OF BARE HULL @ 100 FT/SEC (AS BEFORE)	DH FROM PART 3 BARE HULL DRAG @ 100 FT/SEC (AS BEFORE) (LBS)							
1	84000	4.92428	1.64143	1.79207	1.66667	3.45874	1.81731	3.81731	4.90199	0.1919	730	4.01799	1090	0.1695	1540	4.11834	4.21868	2120	0.1308	5229	0.1280	379207	420	0.1855
2	180000	5.25527	1.75176	1.89259	"	3.55926	1.80750	3.80750	0.1861	710	4.00818	1060	0.1648	1480	4.10853	4.20887	2060	0.1271	4902	0.1200	389259	640	0.176	
3	190000	5.27875	1.75958	1.89927	"	3.56594	1.80636	3.80636	0.1841	705	4.00704	1056	0.1642	1478	4.10739	4.20773	2058	0.1270	4861	0.1190	389927	660	0.175	
4	95000	4.77772	1.65924	1.80218	"	3.46885	1.80961	3.80961	0.1880	715	4.01029	1080	0.1679	1500	4.11064	4.21098	2080	0.1283	5024	0.1229	380218	432	0.176	
5	95000	4.77772	1.65924	1.80218	"	3.46885	1.80961	3.80961	0.1880	715	4.01029	1080	0.1679	1500	4.11064	4.21098	2080	0.1283	5024	0.1229	380218	432	0.176	
6	797000	5.90146	1.96715	2.13711	"	3.80378	1.83663	3.83663	0.2115	805	4.03731	1188	0.1847	1630	4.13766	4.23800	2270	0.1413	5955	0.1359	413711	1650	0.163	
7	2764461	6.44161	2.14720	2.31877	"	3.98544	1.83824	3.83824	0.2115	806	4.03892	1190	0.1850	1640	4.13927	4.23961	2285	0.1409	4743	0.1210	431877	3120	0.125	
8	400000	5.60206	1.86735	2.06034	1.66667	3.72701	1.85966	3.85966	0.2331	896	4.06034	1265	0.1967	1758	4.16069	4.26103	2495	0.1539	5392	0.1320	406034	1275	0.199	
9	572000	5.75740	1.91913	2.10360	"	3.77027	1.85114	3.85114	0.2311	880	4.05182	1230	0.1913	1710	4.15217	4.25251	2422	0.1594	6209	0.1520	410360	1460	0.179	
10	734000	5.86574	1.95525	2.13994	"	3.80661	1.85136	3.85136	0.2315	882	4.05204	1240	0.1928	1710	4.15239	4.25273	2422	0.1594	6291	0.1540	413994	1640	0.171	
11	631000	5.80003	1.93334	2.11788	"	3.78455	1.85121	3.85121	0.2311	880	4.05188	1238	0.1925	1710	4.15224	4.25258	2422	0.1594	6250	0.1530	411786	1530	0.164	
12	744000	5.87157	1.95719	2.13719	"	3.80386	1.84667	3.84667	0.2233	830	4.04735	1228	0.1894	1680	4.14770	4.24804	2384	0.1571	6299	0.1520	413719	1630	0.168	
13	787000	5.89597	1.96532	2.15339	"	3.82006	1.85474	3.85474	0.2323	885	4.05542	1245	0.1936	1718	4.15777	4.25611	2428	0.1498	6454	0.1460	415339	1700	0.167	
14	858000	5.93349	1.97783	2.16407	"	3.83074	1.85291	3.85291	0.2319	883	4.05359	1241	0.1929	1714	4.15374	4.25428	2425	0.1496	6617	0.1467	416407	1750	0.164	
15	884000	5.94645	1.98215	2.16997	"	3.83666	1.85451	3.85451	0.2323	885	4.05519	1245	0.1936	1718	4.15554	4.25588	2425	0.1498	6495	0.1589	416999	1775	0.163	
16	1220000	6.08636	2.02879	2.20659	"	3.87326	1.84447	3.84447	0.2225	848	4.04515	1210	0.1882	1675	4.14550	4.24584	2340	0.1444	6250	0.1530	420659	1770	0.146	
17	1363000	6.13450	2.04483	2.22803	"	3.89470	1.84987	3.84987	0.2241	852	4.05055	1212	0.1885	1700	4.15070	4.23580	2410	0.1487	6168	0.1509	422803	2110	0.145	
18	2147000	6.33224	2.11075	2.28857	"	3.95524	1.84449	3.84449	0.2194	835	4.04517	1211	0.1883	1676	4.14552	4.25174	2340	0.1444	5310	0.1300	428857	2615	0.133	
19	2140000	6.33041	2.11014	2.28813	"	3.95480	1.84466	3.84466	0.2194	835	4.04534	1213	0.1886	1680	4.14567	4.24603	2380	0.1468	5310	0.1300	428813	2615	0.133	
20	2140000	6.33041	2.11014	2.28813	"	3.95480	1.84466	3.84466	0.2194	835	4.04534	1213	0.1886	1680	4.14567	4.24603	2380	0.1468	5310	0.1300	428813	2615	0.133	
21	2141000	6.33062	2.11021	2.28813	"	3.95480	1.84459	3.84459	0.2194	835	4.04527	1213	0.1886	1678	4.14562	4.24603	2378	0.1467	5310	0.1300	428813	2615	0.133	
22	2640000	6.42160	2.14053	2.32599	"	3.97266	1.85213	3.85213	0.2279	866	4.05281	1260	0.1959	1712	4.15316	4.25350	2420	0.1493	5351	0.1310	432599	3050	0.135	
23	2640000	6.42160	2.14053	2.32599	"	3.97266	1.85213	3.85213	0.2279	866	4.05281	1260	0.1959	1712	4.15316	4.25350	2420	0.1493	5351	0.1310	432599	3050	0.135	
24	2141000	6.33062	2.11021	2.28813	"	3.95480	1.84457	3.84457	0.2194	835	4.04527	1210	0.1881	1678	4.14562	4.24576	2360	0.1456	5024	0.1229	428813	2615	0.133	
25	2400000	6.38021	2.12674	2.31589	"	3.98256	1.85582	3.85582	0.2272	865	4.05650	1260	0.1959	1720	4.15686	4.25720	2425	0.1496	5310	0.1300	431589	2905	0.137	
26	2289861	6.35980	2.11993	2.30105	"	3.96772	1.84779	3.84779	0.2233	840	4.04847	1220	0.1897	1695	4.14882	4.24716	2390	0.1454	5147	0.1259	430105	2740	0.134	

CALCULATIONS BY CLINTON H. HAVILL  
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PART II

ITEM	PRELIM. PLOT # 2										COMPARISON OF BARE HULL DRAG AT 100 FT./SEC.									
	U <sub>MAX</sub> FROM PART I FT./SEC	TOTAL AREA OF DRAG WHOLE SHIP POWER ON Sq. Ft.	TOTAL DRAG OF WHOLE SHIP @ U <sub>MAX.</sub> LBS.	% OF TOTAL DRAG FOR HULL (ESTIMATED)	BARE HULL DRAG AT U <sub>MAX.</sub> LBS.	LOG <sub>10</sub> OL AT U <sub>MAX.</sub>	VIRTUAL VOLUME V <sub>M</sub> FROM PART I CU. FT.	LOG <sub>10</sub> V <sub>M</sub>	LOG <sub>10</sub> V <sub>M</sub> LA	LOG <sub>10</sub> V <sub>M</sub> LA = LOG <sub>10</sub> GEOM. LENGTH = LOG <sub>10</sub> L <sub>g</sub>	U <sub>MAX</sub> FROM PART I FT./SEC	LOG <sub>10</sub> U <sub>MAX.</sub> IN PLOTS 1 & 2	LOG <sub>10</sub> U <sub>g</sub> @ U <sub>MAX.</sub> SAME AS USED IN PLOTS 1 & 2	BARE HULL DRAG AT U <sub>MAX.</sub> FROM PLOT 3 LBS	DRAG COEF. OF BARE HULL AT U <sub>MAX.</sub> $\frac{D_H}{(\frac{V_M}{L_A})^2}$	LOG <sub>10</sub> U <sub>g</sub> @ 100 FT./SEC = LOG <sub>10</sub> U <sub>g</sub> + 2	BARE HULL DRAG AT 100 FT./SEC FROM PLOT # 3 LBS	DRAG COEF. OF BARE HULL @ 100 FT./SEC. C <sub>H</sub>		
1	67.00	87.40	493.08	44	216.95	3.63092	90380	4.95607	7.16826	1.79207	67.00	1.83885	3.63092	215	.0199	3.79207	.0185			
2	88.00	127.40	1167.09	47*	549.47	3.83707	189710	5.27809	7.57035	1.89259	88.00	1.94448	3.83707	547	.0186	3.89259	.0170			
3	83.11	131.49	1076.06	48*	516.50	3.81892	199710	5.30040	7.59707	1.89927	83.11	1.91965	3.81892	521	.0193	3.89927	.0169			
4	82.20	78.00	624.53	53	331.00	3.71705	99820	4.99922	7.20874	1.80218	82.20	1.91487	3.71705	331	.0198	3.80218	.0175			
5	77.30	79.01	559.42	57	318.86	3.69036	99820	4.99922	7.20874	1.80218	77.30	1.88818	3.69036	290	.0217	3.80218	.0175			
6	119.99	170.01	2898.50	60	1739.10	4.21625	827920	5.91799	8.54842	2.13711	119.99	2.07914	4.21625	2110	.0160	4.13711	.0163			
7	115.00	356.99	5594.57	64*	3580.52	4.37947	2862061	6.45667	9.27509	2.31877	115.00	2.06070	4.37947	3910	.0118	4.31877	.0125			
8	26.40	251.00	207.26	51	105.70	3.48194	407310	5.60992	8.24136	2.06034	26.40	1.42160	3.48194	106	.0236	4.06034	.0199			
9	41.00	383.01	762.92	41	312.79	3.71638	582200	5.76507	8.41440	2.10360	41.00	1.61278	3.71638	311	.0224	4.10360	.0179			
10	51.98	374.00	1196.61	46	550.44	3.85578	746680	5.87312	8.55976	2.13994	51.98	1.71584	3.85578	556	.0212	4.13994	.0171			
11	62.40	563.00	2597.85	43	1117.07	3.91304	643680	5.80867	8.47153	2.11788	62.40	1.79518	3.91304	765	.0209	4.11788	.0164			
12	67.50	412.00	2224.31	41	911.96	3.96649	759210	5.88036	8.54875	2.13719	67.50	1.82930	3.96649	916	.0206	4.13719	.0168			
13	66.69	482.00	2535.94	38	963.65	3.97745	802210	5.90428	8.61355	2.15339	66.69	1.82406	3.97745	955	.0212	4.15339	.0167			
14	70.89	409.00	2421.52	46	1113.89	4.01465	873210	5.94111	8.65628	2.16407	70.89	1.85058	4.01465	1090	.0204	4.16407	.0164			
15	77.09	333.98	2350.85	54	1269.46	4.05699	703040	5.95569	8.67997	2.16999	77.09	1.88700	4.05699	1274	.0195	4.16999	.0163			
16	81.61	393.00	3101.08	52	1612.56	4.11833	1250900	6.09722	8.82638	2.20659	81.61	1.91174	4.11833	1520	.0179	4.20659	.0146			
17	86.00	369.49	3238.28	55	1781.05	4.16253	1393900	6.14422	8.91212	2.22803	86.00	1.93450	4.16253	1775	.0165	4.22803	.0145			
18	92.40	474.00	4795.13	51	2445.51	4.25424	2211820	6.34473	9.15429	2.28857	92.40	1.96567	4.25424	2425	.0144	4.28857	.0133			
19	92.41	423.00	4279.49	56	2396.51	4.25385	2202820	6.34297	9.15253	2.28813	92.41	1.96972	4.25385	2405	.0144	4.28813	.0133			
20	96.19	372.00	4077.12	63	2568.58	4.27126	2202820	6.34297	9.15253	2.28813	96.19	1.98313	4.27126	2587	.0142	4.28813	.0133			
21	97.40	371.00	4170.78	64	2669.29	4.27669	2203820	6.34297	9.15253	2.28813	97.40	1.98856	4.27669	2661	.0142	4.28813	.0133			
22	94.29	419.00	4413.75	66	2913.07	4.30046	2702820	6.43182	9.30398	2.32599	94.29	1.97447	4.30046	2910	.0144	4.32599	.0135			
23	94.19	424.00	4456.66	65	2896.83	4.29999	2702820	6.43182	9.30398	2.32599	94.19	1.97400	4.29999	2894	.0144	4.32599	.0135			
24	104.81	372.00	4851.25	63	3056.28	4.30853	2203820	6.34297	9.15253	2.28813	104.81	2.02040	4.30853	3067	.0142	4.28813	.0133			
25	113.12	404.00	6123.83	63	3858.01	4.36943	2462820	6.39142	9.26358	2.31789	113.12	2.05354	4.36943	3893	.0143	4.35899	.0137			
26	91.00	402.51	3949.43	62*	2448.64	4.26009	2352681	6.37156	9.20420	2.30105	91.00	1.95904	4.26009	2480	.0143	4.30105	.0134			
					D <sub>H</sub>		V <sub>M</sub>	LA						D <sub>H</sub>	C <sub>H</sub>	D <sub>H</sub>	C <sub>H</sub>			

EXPLANATION

COMPARISON OF SHIPS IF REDUCED OR EXPANDED TO THE SAME AIR VOLUME OF HULL. RELATION BETWEEN TWO SIMILAR SOLIDS OF LINEAR DIMENSIONS IN THE RATIO OF  $\frac{L_1}{L_2}$  ARE AS FOLLOWS: -  $\frac{VOL_1}{VOL_2} = (\frac{L_1}{L_2})^3$ . FOR THE SAME OR GEOMETRICALLY SIMILAR AIRSHIP HULLS BUT OF DIFFERENT SIZES. AIR VOL. OF HULL<sub>1</sub> =  $(\frac{L_1}{L_2})^3$  AIR VOL. OF HULL<sub>2</sub> =  $(\frac{L_2}{L_1})^3$  (GEOMETRIC LENGTH)

ACTUAL AIR VOL. OF HULL =  $(\frac{ACTUAL L_g}{NEW L_g})^3$   
 NEW AIR VOL. OF HULL =  $(\frac{ACTUAL L_g}{NEW L_g})^3$

TO REDUCE ALL ITEMS TO 100,000 CU. FT. AIR VOL. OF HULL,  $\frac{ACTUAL AIR VOL. OF HULL}{100000} = (\frac{ACTUAL L_g}{NEW L_g})^3$  BY LOGS<sub>10</sub>.

$LOG_{10}(NEW L_g) = \frac{1}{3} LOG_{10} 100000 - \frac{1}{3} LOG_{10}(ACTUAL AIR VOL.) + LOG_{10}(ACTUAL L_g)$   
 $LOG_{10}(NEW L_g) = \frac{2}{3} - \frac{1}{3} LOG_{10}(AIR VOL.) + LOG_{10}(ACTUAL L_g)$

