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	TECHNICAL NOTE 3028
	THE COMPRESSIBLE LAMINAR BOUNDARY LAYER WITH HEAT
	TRANSFER AND SMALL PRESSURE GRADIENT
	By George M. Low
	Lewis Flight Propulsion Laboratory Cleveland, Ohio
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THE COMPRESSIBLE LAMINAR BOUNDARY LAYER WITH HEAT TRANSFER

AND SMALL PRESSURE GRADIENT

By George M. Low

SUMMARY

A perturbation method for the calculation of velocity and temperature profiles and skin-friction and heat-transfer characteristics for two-dimensional, compressible laminar boundary layers with heat transfer and a small arbitrary pressure gradient is presented. The permissible pressure gradients include those of a form and magnitude usually encountered over slender aerodynamic shapes in supersonic flight. The method applies for any constant Prandtl number, but results, aside from special examples, are presented for a Prandtl number of 0.72. For the case of heat transfer, the wall temperature is assumed constant.

A large number of universal functions are given in tabular form, so that the amount of effort required in a practical application is reduced to the arithmetic combination of several tabulated values. The computation procedure is summarized in a section entitled "APPLICATION OF ANALYSIS."

The combined effects of heat transfer and pressure gradient on boundary-layer characteristics are demonstrated by applying the results of the analysis to two representative wings.

INTRODUCTION

Interest in the characteristics of the laminar boundary layer has increased in recent years because, under certain conditions, the boundary layer may remain laminar over large areas of airplanes and missiles. For example, Van Driest (ref. 1) has shown theoretically that if the solid boundary is cooled sufficiently, the laminar boundary layer can be stabilized regardless of Reynolds number at Mach numbers between 1 and 9. Sternberg (ref. 2) observed laminar boundary layers at Reynolds numbers as high as 50×10^6 in flight tests of the V-2 rocket. Laminar boundary layers may also be expected in flight at very high altitude where the density, and hence the Reynolds number per unit length, will be low.

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Solutions of the compressible laminar-boundary-layer equations for the special case of zero pressure gradient have been obtained by several authors. The theory of Chapman and Rubesin (ref. 3), for example, presents a very simple method for calculating boundary-layer characteristics over a flat plate with arbitrary heat transfer. The more recent, and in general more exact, studies of Klunker and McLean (refs. 4 and 5), Van Driest (ref. 6), Young and Janssen (ref. 7), and Moore (ref. 8) have demonstrated that the theory of Chapman and Rubesin yields excellent results for reasonably low ambient air temperatures at Mach numbers up to about 5.

Solutions for the more general case of arbitrary heat transfer and arbitrary pressure gradient are still in an early stage of development. Tani, in a little known paper (ref. 9), used a perturbation procedure to obtain direct solutions of the boundary-layer differential equations with a Falkner-Skan type external velocity distribution ($u_{e} \sim x^{m}$) and heat transfer. Results are easily obtainable from tabulated functions, but are limited to a Prandtl number of 1, small Mach numbers, and small rates of heat transfer. Furthermore, the Falkner-Skan type of external velocity distribution is not appropriate for supersonic flow over thin wings. Ginzel (ref. 10), Kalikhman (ref. 11), and Libby and Morduchow (extension of ref. 12) have obtained solutions of the compressible laminar-boundary-layer equations by an extended Pohlhausen method. However, the accuracy of the Pohlhausen method under conditions of heat transfer at high speeds has not been determined. In addition, the amount of work required in a particular application of references 10 and 11 is prohibitive because the simultaneous numerical solution of two differential equations is required. Libby and Morduchow avoid this difficulty by the additional assumption that certain variable quantities remain constant over the entire length of boundary-layer development.

The purpose of the present report is to present a method of solution developed at the NACA Lewis laboratory that is free of many of the limitations of references 9 to 12. An accurate method for calculating velocity and temperature profiles and skin-friction and heat-transfer characteristics for the compressible laminar boundary layer with heat transfer and a small pressure gradient is derived. The permissible pressure gradient may be of a form and magnitude usually encountered over thin aerodynamic shapes in supersonic flight. The solution is obtained by a method of perturbation on the flat-plate solution of Chapman and Rubesin; it constitutes the first two terms of a Maclaurin series expansion in terms of the free-stream velocity gradient parameter. The method involves the direct solution of the boundary-layer differential equations. Although the theory applies for any 'constant Prandtl number, tabulated results presented in this report apply, in general, for a Prandtl number of 0.72. For the case of heat transfer, results are limited to an isothermal wall.

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Solutions of the first-order perturbation equations are presented in tabular form, so that the amount of effort required in a particular application is reduced to the arithmetic combination of several tabulated values. A section of the report entitled "APPLICATION OF ANALYSIS" is included in order to facilitate the application of results in practical applications.

ASSUMPTIONS AND LIMITATIONS

The following simplifying assumptions and limitations are imposed in addition to the usual boundary-layer assumptions:

(1) The ratio of the velocity at the outer edge of the boundary layer u_e to a reference velocity u_r can be represented by

$$\frac{u_e}{u_r} = 1 + \varepsilon a_N x^N \tag{1}$$

where the repeated index N indicates a summation over several values of N. (All symbols used in this report are defined in appendix A.) In equation (1) the quantity $a_N x^N$ represents the shape of the deviation of u_e from a constant value, while ϵ represents the magnitude of this deviation. The quantity ϵ is assumed small as compared with unity, whereas the quantity $a_N x^N$ is of normal order of magnitude. This type of external velocity distribution is capable of representing in form and magnitude those encountered over thin aerodynamic shapes at Mach numbers greater than 1.

(2) The temperature of the solid boundary is constant under conditions of heat transfer. (Under conditions of zero heat transfer the wall temperature will be a calculated function of the pressure distribution.)

(3) The viscosity and temperature are related linearly by the following expression:

$$\frac{\mu}{\mu_r} = C \frac{t}{t_r}$$
(2)

Chapman and Rubesin have shown that solutions of the boundary-layer equations based on equation (2) agree well with more exact solutions for flat-plate flows at Mach numbers less than 5 if the constant C is determined by matching equation (2) with Sutherland's relation at the solid boundary

$$C = \sqrt{\frac{t_w}{t_r}} \frac{(t_r + S)}{(t_w + S)}$$
(3)

This assumption should also be reasonable for flows with slight streamwise pressure gradients, especially when the wall temperature is constant. For a nonisothermal wall an average wall temperature should be used in equation (3), as suggested in reference 3.

(4) The Prandtl number and specific heat are constant throughout the boundary layer. The restriction imposed by this assumption is not great because both Pr and c_p vary only slightly at moderate temperatures. A Prandtl number of 0.72 was used in all calculations.

GOVERNING EQUATIONS

Differential equations and boundary conditions. - The equations governing the steady laminar flow of a viscous compressible fluid in a thin boundary layer are the momentum equations

$$uu_{\mathbf{x}} + vu_{\mathbf{y}} = -\frac{1}{\rho} p_{\mathbf{x}} + \frac{1}{\rho} (\mu u_{\mathbf{y}})_{\mathbf{y}}$$
(4a)

$$p_{y} = 0 \tag{4b}$$

the equation of continuity

$$(\rho u)_{x} + (\rho v)_{y} = 0$$
 (5)

the energy equation

$$\rho c_p(ut_x + vt_y) = up_x + (kt_y)_y + \mu(u_y)^2$$
(6)

and the equation of state

$$p = \rho R t \tag{7}$$

The following boundary conditions are imposed on the momentum and energy equations:

$$u(x,0) = 0 u(x,\infty) = u_e$$

$$v(x,0) = 0 t(x,\infty) = t_e$$

$$t(x,0) = t_w (heat transfer)$$

$$t_v(x,0) = 0 (zero heat transfer) (8)$$

At the outer edge of the boundary layer, velocity and pressure are related by the Bernoulli equation, which is

$$\frac{\mathrm{d}\mathbf{p}_{\mathrm{e}}}{\mathrm{d}\mathbf{x}} = -\rho_{\mathrm{e}}\mathbf{u}_{\mathrm{e}} \frac{\mathrm{d}\mathbf{u}_{\mathrm{e}}}{\mathrm{d}\mathbf{x}} \tag{9}$$

The energy equation that applies at the outer edge of the boundary layer is

$$c_p T = c_p t_e + \frac{u_e^2}{2}$$
 (10)

Transformation of Howarth. - In reference 13 Howarth introduced a transformation which, when applied to the momentum and energy equations, yields equations similar in form to the incompressible-boundary-layer equations. First, it is convenient to introduce the dimensionless variables

$$p^{*} = p/p_{r} \qquad u^{*} = u/u_{r} \\ t^{*} = t/t_{r} \qquad v^{*} = v/u_{r} \\ \rho^{*} = \rho/\rho_{r} \qquad \mu^{*} = \mu/\mu_{r}$$
 (11)

Howarth's transformation proceeds as follows: The independent variables x and y are related to the variables x and n according to the following transformation:

$$x \equiv x$$

$$n \equiv \sqrt{\frac{p^{*}}{C}} \int_{0}^{y} \frac{1}{t^{*}} dy$$
(12)

where n distorts the scale in the direction normal to the surface. The derivatives with respect to x and y can be expressed as

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$$\frac{\partial}{\partial x}\Big|_{y} = \frac{\partial}{\partial x}\Big|_{n} + \frac{\partial n}{\partial x}\frac{\partial}{\partial n} \\
\frac{\partial}{\partial y} = \sqrt{\frac{p^{*}}{c}}\frac{1}{t^{*}}\frac{\partial}{\partial n}$$
(13)

Equation (5) is satisfied by a stream function defined as follows:

 $\rho^{*} u^{*} = \psi_{y}$ $\rho^{*} v^{*} = - \psi_{x}$

The stream function $\psi(x,y)$ can be related to a transformed function $\varphi(x,n)$ by

$$\psi(\mathbf{x},\mathbf{y}) \equiv \sqrt{Cp^*} \phi(\mathbf{x},\mathbf{n}) \tag{14}$$

The velocity components now become

Substitution of equations (2), (7), (9), and (12) to (15) into equation (4a) yields the momentum equation in the transformed (x,n) plane

$$\varphi_{n} \varphi_{nx} - \varphi_{x} \varphi_{nn} - \frac{\nu_{r}}{u_{r}} \varphi_{nnn} = \frac{u_{e}^{*}}{t_{e}^{*}} \frac{du_{e}^{*}}{dx} \left[t^{*} - \frac{1}{2} \gamma M_{r}^{2} \varphi \varphi_{nn} \right]$$
(16)

The energy equation in the x,n-plane becomes

The following boundary conditions apply to equations (16) and (17):

 $\varphi(x,0) = 0 \qquad \qquad \varphi_n(x,\infty) = u_e^*$ $\varphi_n(x,0) = 0$ $t^*(x,0) = t_w^* \text{ (heat transfer)}$ $t_n^*(x,0) = 0 \text{ (zero heat transfer)}$ $t^*(x,\infty) = t_e^*$

PERTURBATION ANALYSIS

Expansion of momentum and energy equations in powers of ε . - For the special case of $u_{e}^{*} = 1$ (zero pressure gradient), equations (16) and (17) become identical to the momentum and energy equations solved by Chapman and Rubesin. It therefore appears logical to let u_{e}^{*} differ from unity by a small amount in order to obtain a perturbation solution for flows with small pressure gradients. As discussed under ASSUMPTIONS AND LIMITATIONS, the external velocity at the outer edge of the boundary layer is taken to be of the following form:

$$u_e^* = 1 + \varepsilon a_N x^N \tag{1}$$

Substitution of equations (1) and (7) into equations (9) and (10) and elimination of higher-order terms yield:

$$p^* = 1 - \gamma M_r^2 \varepsilon a_N x^N$$
 (18)

and

$$t_{e}^{*} = 1 - (\gamma - 1) M_{r}^{2} \varepsilon a_{N} x^{N}$$
(19)

Within the boundary layer the stream function and temperature are replaced by their Maclaurin series expansions in terms of the velocity gradient parameter ε :

$$\varphi(\mathbf{x},\mathbf{n},\varepsilon) = \overline{\varphi}(\mathbf{x},\mathbf{n}) + \varepsilon \mathbf{a}_{\mathrm{N}} \,\overline{\overline{\varphi}}_{\mathrm{N}}(\mathbf{x},\mathbf{n}) + \varepsilon^2 \mathbf{a}_{\mathrm{NM}} \,\overline{\overline{\varphi}}_{\mathrm{NM}}(\mathbf{x},\mathbf{n}) + \ldots \quad (20)$$

$$t^{*}(x,n,\varepsilon) = \overline{t}(x,n) + \varepsilon a_{\mathbb{N}} \overline{\overline{t}}_{\mathbb{N}}(x,n) + \varepsilon^{2} a_{\mathbb{N}} \overline{\overline{\overline{t}}}_{\mathbb{N}}(x,n) + \dots \quad (21)$$

A sequence of momentum and energy equations is obtained by substitution of equations (1) and (18) to (21) into equations (16) and (17), and by equating coefficients of like powers of ε . The zero-order equations, obtained by equating coefficients of (ε)⁰, are:

$$\overline{\phi}_{n} \,\overline{\phi}_{nx} - \overline{\phi}_{x} \,\overline{\phi}_{nn} - \frac{\nu_{r}}{u_{r}} \,\overline{\phi}_{nnn} = 0$$
(22)

$$\overline{\varphi}(\mathbf{x},0) = \overline{\varphi}_{\mathbf{n}}(\mathbf{x},0) = 0 \qquad \overline{\varphi}_{\mathbf{n}}(\mathbf{x},\infty) = 1$$

and

$$\overline{\phi}_{n} \overline{t}_{x} - \overline{\phi}_{x} \overline{t}_{n} - \frac{\nu_{r}}{u_{r} Pr} \overline{t}_{nn} = (\gamma - 1) \frac{\nu_{r} M_{r}^{2}}{u_{r}} (\overline{\phi}_{nn})^{2}$$

$$\overline{t}(x,0) = t_{W}^{*} \text{ (heat transfer)}$$
(23)

 $\overline{t}_n(x,0) = 0$ (zero heat transfer)

<u>t</u>(x,∞) = 1

Equating coefficients of ϵ yields the first-order equations:

$$\overline{\phi}_{n} \ \overline{\phi}_{Nnx} + \overline{\phi}_{nx} \ \overline{\phi}_{Nn} - \overline{\phi}_{x} \ \overline{\phi}_{Nnn} - \overline{\phi}_{nn} \ \overline{\phi}_{Nx} - \frac{\nu_{r}}{u_{r}} \ \overline{\phi}_{Nnnn}$$

$$= \mathbb{N} \ x^{N-1} \ (\overline{t} - \frac{\gamma}{2} \ M_{r}^{2} \ \overline{\phi} \ \overline{\phi}_{nn}) \ (24)$$

$$\overline{\phi}_{N}(x,0) = \overline{\phi}_{Nn}(x,0) = 0 \qquad \overline{\phi}_{Nn}(x,\infty) = x^{N}$$

and

$$\overline{\overline{\phi}}_{Nn} \overline{t}_{x} + \overline{\phi}_{n} \overline{t}_{Nx} - \overline{\phi}_{Nx} \overline{t}_{n} - \overline{\phi}_{x} \overline{t}_{Nn} - \frac{\nu_{r}}{u_{r} Pr} \overline{t}_{Nnn}$$

$$= 2 \frac{\nu_{r}}{u_{r}} (\gamma - 1) M_{r}^{2} \overline{\phi}_{nn} \overline{\phi}_{Nnn} - Nx^{N-1} M_{r}^{2} \left[\frac{\gamma}{2} \overline{\phi} \overline{t}_{n} + (\gamma - 1) \overline{t} \overline{\phi}_{n} \right]$$

$$= 2 \frac{\nu_{r}}{u_{r}} (\gamma - 1) M_{r}^{2} \overline{\phi}_{nn} \overline{\phi}_{Nnn} - Nx^{N-1} M_{r}^{2} \left[\frac{\gamma}{2} \overline{\phi} \overline{t}_{n} + (\gamma - 1) \overline{t} \overline{\phi}_{n} \right]$$

$$= 2 \frac{\nu_{r}}{t_{N}} (\gamma - 1) M_{r}^{2} \overline{\phi}_{nn} \overline{\phi}_{Nnn} - Nx^{N-1} M_{r}^{2} \left[\frac{\gamma}{2} \overline{\phi} \overline{t}_{n} + (\gamma - 1) \overline{t} \overline{\phi}_{n} \right]$$

$$= 2 \frac{\nu_{r}}{t_{N}} (\gamma - 1) M_{r}^{2} \overline{\phi}_{nn} \overline{\phi}_{Nnn} - Nx^{N-1} M_{r}^{2} \left[\frac{\gamma}{2} \overline{\phi} \overline{t}_{n} + (\gamma - 1) \overline{t} \overline{\phi}_{n} \right]$$

$$= 0 \text{ or } \overline{t}_{Nn} (\gamma - 1) = 0 \qquad \overline{t}_{Nn} (\gamma - 1) \sqrt{N} M_{r}^{2}$$

The higher-order equations are obtained by equating coefficients of ε^2 , ε^3 , and so forth. If it is assumed that the functions $\overline{\phi}$, $\overline{\phi}$, $\overline{\phi}$,

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and so forth, and the functions \overline{t} , \overline{t} , \overline{t} , and so forth, are of the same order of magnitude, then, since ε is postulated to be small, all additional contributions will be of second or higher order and hence may be neglected in a first-order treatment. Further justification for neglecting the higher-order equations comes from reference 14, where it is shown that for incompressible flow the function $\overline{\phi}$ is numerically much smaller than $\overline{\phi}$ and $\overline{\phi}$. The second-order terms therefore should contribute very little to the solution of the boundary-layer equations for flows with small pressure gradients.

The permissible magnitude of the pressure gradient depends largely on the length of run over which it acts. For example, a very small pressure gradient can cause laminar separation if it acts over a large distance. The present method can be applied only if all the deviations in boundary-layer characteristics caused by the pressure gradient are small.

The only dependent variable appearing in equation (22) is $\overline{\phi}$, so that the solution of this equation is independent of all following equations. Furthermore, each succeeding equation involves only one new dependent variable, so that each equation can be solved in principle once the preceding equations have been solved. Equations (22) to (25) still contain two independent variables, however, and require further reduction to make them amenable to solution.

Solution of zero-order equation. - The zero-order equations may be transformed to ordinary differential equations by introduction of the Blasius characteristic variable η :

$$\eta \equiv \frac{n}{2} \sqrt{\frac{u_r}{v_r x}}$$
(26)

The stream function $\overline{\phi}(x,n)$ is related to a new function $f(\eta)$ as follows:

$$\overline{\varphi}(\mathbf{x},\mathbf{n}) = \sqrt{\frac{\mathbf{v}_{\mathbf{r}} \cdot \mathbf{x}}{\mathbf{u}_{\mathbf{r}}}} f(\eta)$$
(27)

The temperature in the x,n-coordinate system is equal to the temperature in the η -system

$$\overline{t}(\mathbf{x},\mathbf{n}) = \overline{t}(\eta)$$
 (28)

With the aid of equations (26), (27), and (28), equations (22) and (23) can be written

$$f''' + ff'' = 0$$
 (29)

$$\overline{t}'' + \Pr f \overline{t}' = - (\Pr) \left(\frac{\gamma - 1}{4}\right) M_r^2 (f'')^2$$
(30)

where the primes indicate differentiation with respect to $\ \eta.$ The boundary conditions of f and t are

$$f(0) = 0$$
 $f'(\infty) = 2$
 $f'(0) = 0$
 $\overline{t}(0) = t_W^*$ or $\overline{t}'(0) = 0$
 $\overline{t}(\infty) = 1$

Equation (29) is the well-known Blasius equation which has been solved by several investigators. In order to eliminate M_r^2 as a parameter in the solution of equation (30), this equation is split into two parts in the following manner:

$$\overline{t}(\eta) = 1 + \frac{\gamma - 1}{2} M_r^2 r(\eta) + K s(\eta)$$
(31)

where $r(\eta)$ and $s(\eta)$ satisfy the following equations:

$$r'' + Pr f r' = -\frac{Pr}{2} (f'')^2$$
 (32)

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$$s'' + \Pr f s' = 0$$
 (33)

The following boundary conditions are applied to equations (32) and (33):

$$r'(0) = 0$$
 $r(\infty) = 0$
 $s'(0) = - [f''(0)]^{Pr}$ $s(\infty) = 0$

Although the solution of equations (32) and (33) can be written in terms of quadratures, as shown in reference 3, the numerical solution of the differential equations was found more convenient. Numerical solutions, as discussed in appendix B, were made by Lynn U. Albers.

The functions f''(0), r(0), s(0), and s'(0) are listed in table I; all other functions resulting from the solution of the zeroorder equations can be found in table II. The constant K (eq. (31)) is related to the rate of heat transfer and hence to the wall temperature. Its value is determined by solving equation (31) at $\eta = 0$.

$$K = \frac{1}{s(0)} \left[(\bar{t}_{w} - 1) - \frac{\gamma - 1}{2} M_{r}^{2} r(0) \right]$$
(34)

where the wall temperature ratio $\overline{t}_w = \frac{t_w}{t_r}$ is, in general, prescribed. For the case of zero heat transfer, K vanishes.

Solution of first-order equations. - The first-order momentum and energy equations can also be transformed to ordinary differential equations with the aid of the Blasius variable together with the following functions:

$$\overline{\overline{\phi}}_{N}(x,n) = 2 \sqrt{\frac{\nu_{r} x}{u_{r}}} x^{N} g_{N}(\eta)$$
(35)

and

$$\vec{t}_{N}(x,n) = - (\gamma-1) M_{r}^{2} x^{N} h_{N}(\eta) \text{ (zero heat transfer)} (36)$$

$$\overline{\overline{t}}_{N}(x,n) = -(\gamma-1) M_{r}^{2} x^{N} H_{N}(\eta) \text{ (heat transfer)}$$
(37)

Equations (24) and (25) are now written

$$g_{N}^{'''} + f g_{N}^{''} - 2N f' g_{N}^{'} + (2N + 1) f'' g_{N}$$
$$= -4N \left\{ 1 + M_{r}^{2} \left[\left(\frac{\gamma - 1}{2} \right) r - \frac{\gamma}{8} f f'' \right] + K_{8} \right\}$$
(38)

and

$$= \Pr\left[\frac{4N+2}{(\gamma-1)M_r^2} g_N \overline{t}' + f'' g_N'' - \frac{\gamma}{\gamma-1} N f\overline{t}' - 2Nf'\overline{t}\right]$$
(39)

The function $H_N(\eta)$ satisfies the same equation as is satisfied by $h_N(\eta)$ (eq. (41)), but is subject to different boundary conditions. The boundary conditions are

$$\begin{split} g_{\overline{N}}(0) &= 0 \qquad g_{\overline{N}}^{i}(\infty) = 1 \\ g_{\overline{N}}^{i}(0) &= 0 \\ h_{\overline{N}}^{i}(0) &= 0 \qquad h_{\overline{N}}(\infty) = 1 \quad (\text{zero heat transfer}) \\ H_{\overline{N}}(0) &= 0 \qquad H_{\overline{N}}(\infty) = 1 \quad (\text{heat transfer}) \end{split}$$

The solution of equation (38) can be obtained in closed form for the special cases of $N = -\frac{1}{2}$ (ref. 15) and N = 0 (see appendix C). In the general case, however, the equation was solved numerically. In order to obtain a numerical solution which applies over a range of Mach numbers and heat-transfer rates, the parameters M_r and K were eliminated from equation (38) by splitting the function g into a linear combination of three independent functions:

$$g_{N} = g_{N1} + M_{r}^{2} g_{N2} + K g_{N3}$$
 (40)

where the three new functions satisfy the following equations:

$$g_{N1}^{""} + fg_{N1}^{"} - 2Nf'g_{N1}^{'} + (2N + 1) f''g_{N1} = -4N$$
 (41)

$$g_{N2}^{""} + fg_{N2}^{"} - 2Nf'g_{N2}' + (2N + 1) f''g_{N2} = \frac{N}{2} [\gamma ff'' - 4(\gamma - 1)r]$$
 (42)

$$g_{N3}^{""} + fg_{N3}^{"} - 2Nf'g_{N3}' + (2N + 1) f''g_{N3} = -4Ns$$
 (43)

$$g_{Ni}(0) = g_{Ni}'(0) = 0 \qquad (i = 1, 2, 3)$$
$$g_{N1}'(\infty) = 1 \qquad g_{N2}'(\infty) = g_{N3}'(\infty) = 0$$

The first-order energy equation can be solved in closed form for $N = -\frac{1}{2}$ (ref. 15) and N = 0 (appendix C). For zero heat transfer, a closed form solution of equation (39) for Pr = 1 can also be obtained for all values of N, as shown in appendix D. For other cases the solution of equation (39) is again found numerically after several parameters have been eliminated by replacing the equation by the following system:

$$h_{N} = h_{NL} + M_{r}^{2} h_{N2}$$
(44)

where

$$h_{Nl}^{"} + \Pr f h_{Nl}^{'} - 2\Pr Nf' h_{Nl} = \Pr \left[(2N + 1)g_{Nl}r' + f''g_{Nl}'' - 2Nf' \right]$$
(45)

$$h_{N2}'' + Pr fh_{N2}' - 2Pr Nf'h_{N2}$$

$$= \Pr\left[(2N + 1)g_{N2} r' + f''g_{N2}'' - \frac{\gamma Nr'f}{2} - N(\gamma - 1)rf'\right]$$
(46)

$$h_{N1}^{t}(0) = 0$$
 $h_{N1}(\infty) = 1$ $h_{N2}(\infty) = 0$ (i = 1,2)

Equations (44) to (46) apply for the case of zero heat transfer. For flows with heat transfer, the following equations are obtained:

$$H_{N} = H_{N1} + M_{r}^{2} H_{N2} + KH_{N3} + \frac{K}{M_{r}^{2}} H_{N4} + \frac{K^{2}}{M_{r}^{2}} H_{N5}$$
(47)

The functions $H_{N1}(\eta)$ and $H_{N2}(\eta)$ satisfy the same equations or are satisfied by $h_{N1}(\eta)$ and $h_{N2}(\eta)$, but are subject to modified boundary conditions. The remaining functions in equation (47) satisfy the following equations:

$$H_{N3}'' + \Pr f H_{N3}' - 2\Pr Nf' H_{N3} = \Pr \left[(2N + 1) \left(\frac{2g_{N2}s'}{\gamma - 1} + r' g_{N3} \right) + f'' g_{N3}'' - \frac{\gamma Nfs'}{\gamma - 1} - 2Nf's \right]$$
(48)

$$H_{N4}'' + \Pr fH_{N4}' - 2\Pr Nf'H_{N4} = \Pr \left[(2N + 1) \left(\frac{2g_{N1}s'}{\gamma - 1} \right) \right]$$
(49)

$$H_{N5}^{"} + \Pr f H_{N5}^{'} - 2\Pr Nf' H_{N5} = \Pr \left[(2N + 1) \left(\frac{2g_{N3} s'}{\gamma - 1} \right) \right]$$
(50)
$$H_{N1}(0) = 0 \qquad H_{N1}(\infty) = 1 \qquad (i = 1, 2, 3, 4, 5)$$

$$\mathbb{H}_{\mathbb{N}\mathbb{Z}}(\infty) = \mathbb{H}_{\mathbb{N}\mathbb{J}}(\infty) = \mathbb{H}_{\mathbb{N}\mathbb{4}}(\infty) = \mathbb{H}_{\mathbb{N}\mathbb{5}}(\infty) = 0$$

The elimination of M_r and K as parameters in the first-order equations has thus yielded ten equations for each value of N. These

equations were solved numerically for N = 1, 2, and 3, as discussed in appendix B. The functions g_N^{I} , which represent the first-order velocity corrections, and h_N and H_N , which represent the first-order temperature corrections, are presented graphically in figures 1, 2, and 3. Tabulated results of all solutions of the first-order equations can be found in tables III, IV, and V. Initial values are also tabulated in table I.

The solutions of the zero- and first-order equations can now be combined to yield velocity and temperature profiles, skin-friction and heat-transfer coefficients, recovery factors, and displacement thickness.

BOUNDARY-LAYER CHARACTERISTICS

Velocity and temperature profiles. - The dimensionless velocity u* is related to the characteristic variable η through equations (15), (20), (27), and (35) in the following manner:

$$\mathbf{u}^* \cong \frac{1}{2} \mathbf{f}'(\eta) + \varepsilon \mathbf{a}_{\mathbf{N}} \mathbf{x}^{\mathbf{N}} \mathbf{g}'_{\mathbf{N}}(\eta)$$
 (51)

From equations (21), (28), (31), and (36) or (37), the temperature profile can be expressed in terms of η as

$$t^* \cong 1 + \frac{\gamma - 1}{2} M_r^2 \left[r(\eta) - 2\varepsilon a_N x^N h_N(\eta) \right]$$
 (52)

for the case of zero heat transfer. For flows with arbitrary heat transfer, the following expression applies:

$$t^* \cong 1 + K \mathfrak{s}(\eta) + \frac{\gamma - 1}{2} M_r^2 \left[\mathbf{r}(\eta) - 2\varepsilon \mathbf{a}_N \mathbf{x}^N \mathbf{H}_N(\eta) \right]$$
(53)

Equations (51), (52), and (53) represent the velocity and temperature profiles as functions of η . The transformation to the physical (x,y-) plane, according to equations (12) and (26), is

$$y = 2 \sqrt{\frac{\nu_r x C}{p^* u_r}} \int_0^{\eta} t^* d\eta$$
 (54)

The value of t^* can be obtained from equations (52) or (53), while p^* is given by equation (18). Thus, for zero heat transfer,

$$y \approx 2 \sqrt{\frac{\nu_r \text{ xC}}{u_r}} \left\{ \left[1 + \frac{\gamma}{2} M_r^2 \varepsilon a_N x^N \right] \left[\eta + \frac{\gamma - 1}{2} M_r^2 \text{ Ir}(\eta) \right] - (\gamma - 1) M_r^2 \varepsilon a_N x^N \text{Ih}_N \right\}$$
(55)

and for arbitrary rates of heat transfer,

$$y \cong \frac{2}{\cdot} \sqrt{\frac{\nu_{r} \text{ xC}}{u_{r}}} \left\{ \left[1 + \frac{\gamma}{2} M_{r}^{2} \varepsilon a_{N} x^{N} \right] \left[\eta + \frac{\gamma - 1}{2} M_{r}^{2} \text{ Ir}(\eta) + \text{KI s}(\eta) \right] - (\gamma - 1) \varepsilon a_{N} x^{N} M_{r}^{2} \text{ IH}_{N}(\eta) \right\}$$
(56)

Skin friction and heat transfer. - The shearing stress at a point in the boundary layer can be obtained from equations (2), (13), and (15):

$$\tau \equiv \mu \frac{\partial u}{\partial y} = \mu_r u_r \sqrt{C p^*} \phi_{nn}$$
 (57)

In terms of the Blasius variable η and after substitution of equation (18) for p^* , equation (57) becomes

$$\tau \simeq \frac{u_r}{4} \sqrt{\frac{\mu_r \rho_r u_r C}{x}} \left\{ f''(\eta) + 2 \varepsilon a_N x^N \left[g_N''(\eta) - \frac{\gamma}{4} M_r^2 f''(\eta) \right] \right\}$$
(58)

Local and average skin-friction coefficients are obtained from the wall shearing stress and the following respective definitions:

$$c_{f} = \frac{\tau_{w}}{\frac{1}{2} \rho_{r} u_{r}^{2}}$$
(59)

and

$$C_{\rm F} = \frac{1}{\frac{1}{2} \rho_{\rm r} u_{\rm r}^2 x} \int_0^{\rm x} \tau_{\rm w} \, dx$$
 (60)

A local skin-friction parameter is obtained from equations (58) and (59):

$$c_{f}\sqrt{\frac{\mathrm{Re}}{\mathrm{C}}} \approx \frac{1}{2} \left\{ f''(0) + 2\varepsilon a_{N}x^{N} \left[g_{N}''(0) - \frac{\gamma}{4} M_{r}^{2} f''(0) \right] \right\}$$
(61)

The average friction drag parameter, obtained from equations (58) and (60), is

$$C_{\mathbf{F}}\sqrt{\frac{\mathrm{Re}}{\mathrm{C}}} \cong \mathbf{f}^{"}(\mathbf{0}) + \frac{2\varepsilon \ \mathbf{a}_{\mathrm{N}}\mathbf{x}^{\mathrm{N}}}{2\mathrm{N}+1} \left[\mathbf{g}_{\mathrm{N}}^{"}(\mathbf{0}) - \frac{\gamma}{4} \ \mathbf{M}_{\mathbf{r}}^{\mathrm{Z}} \ \mathbf{f}^{"}(\mathbf{0}) \right]$$
(62)

The local rate of heat transfer from the surface is given by

$$q \equiv -k \frac{\partial t}{\partial y} \bigg|_{W} = -\frac{c_{p} t_{r} \mu_{r}}{Pr} \sqrt{c_{p} *} t_{n}^{*} \bigg|_{W}$$
$$\equiv -\frac{c_{p}}{2} \frac{c_{p}}{Pr} t_{r} \sqrt{\frac{\mu_{r} \rho_{r} u_{r} C}{x}} \left\{ K s'(0) - (\gamma-1) M_{r}^{2} \varepsilon a_{N} x^{N} \left[H_{N}^{*}(0) + \frac{\gamma}{2(\gamma-1)} K s'(0) \right] \right\}$$
(63)

A dimensionless heat-transfer parameter can now be written as follows:

$$\frac{\mathrm{Nu}}{\sqrt{\mathrm{C} \mathrm{Re}}} \cong \frac{1}{2(\mathrm{t}_{\mathrm{aw}}^{*} - \mathrm{t}_{\mathrm{w}}^{*})} \left\{ \mathrm{K} \mathrm{s}^{*}(0) - (\gamma-1) \mathrm{M}_{\mathrm{r}}^{2} \mathrm{\epsilon} \mathrm{a}_{\mathrm{N}} \mathrm{x}^{\mathrm{N}} \left[\mathrm{H}_{\mathrm{N}}^{*}(0) + \frac{\gamma}{2(\gamma-1)} \mathrm{K} \mathrm{s}^{*}(0) \right] \right\}$$
(64)

where the dimensionless adiabatic wall temperature is

$$\mathbf{t}_{aw}^{*} \cong \mathbf{l} + \frac{\gamma - \mathbf{l}}{2} \, \mathbf{M}_{r}^{2} \left[\mathbf{r}(0) - 2\varepsilon \, \mathbf{a}_{N} \mathbf{x}^{N} \, \mathbf{h}_{N}(0) \right]$$
(65)

<u>Temperature recovery factor.</u> - The temperature recovery factor is derived from equations (19) and (65):

$$F_{R} \equiv \frac{t_{aw}^{*} - t_{e}^{*}}{T^{*} - t_{e}^{*}} \cong r(0) + 2\varepsilon a_{N}x^{N} \left[1 - h_{N}(0) - r(0)\right]$$
(66)

It is evident from the computed results that $h_{N1}(0)$ varies very little with N and is approximately equal to 1 - r(0). With the aid of equation (44), equation (66) is therefore reduced to the following:

$$\mathbf{F}_{\mathrm{R}} \cong \mathbf{r}(0) - 2\varepsilon \, \mathbf{a}_{\mathrm{N}} \mathbf{x}^{\mathrm{N}} \, \mathbf{M}_{\mathrm{r}}^{2} \, \mathbf{h}_{\mathrm{N2}}(0) \tag{67}$$

Displacement thickness. - The boundary-layer displacement thickness is, by definition,

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$$\delta^{*} \equiv \int_{0}^{\infty} \left(1 - \frac{\rho u}{\rho_{e} u_{e}} \right) dy$$
 (68)

$$= 2 \sqrt{\frac{\nabla v_r x}{p^* u_r}} \int_0^{\infty} \left(t^* - \frac{t^*_e}{u^*_e} u^* \right) d\eta$$
 (69)

With the appropriate expressions for p^* , t^* , and u^* , equation (69) becomes, for flows with heat transfer,

$$\delta^{*} \approx 2 \sqrt{\frac{C \nu_{r} x}{u_{r}}} \int_{0}^{\infty} \left\{ \left[1 + \frac{\gamma}{2} M_{r}^{2} \varepsilon a_{N} x^{N} \right] \left[\left(1 - \frac{1}{2} f^{*} \right) + Ks + \frac{\gamma - 1}{2} M_{r}^{2} r \right] + \varepsilon a_{N} x^{N} \left[\frac{1}{2} f^{*} - g_{N}^{*} \right] + \frac{\gamma - 1}{2} M_{r}^{2} \varepsilon a_{N} x^{N} \left[f^{*} - 2H_{N} \right] \right\} d\eta$$
(70)

Integration of equation (70) yields, for Pr = 0.72,

$$\delta^* \cong \sqrt{\frac{C \nu_r x}{u_r}} \left\{ \left[1 + \frac{\gamma}{2} M_r^2 \epsilon a_N x^N \right] \left[1.7208 + 4.0218 K + 1.1094 (\gamma - 1) M_r^2 \right] + \epsilon a_N x^N \left[\alpha_N + (\gamma - 1) M_r^2 B_N \right] \right\}$$
(71)

For the case of zero heat transfer K will vanish and B_N is replaced by β_N . The relation between the functions appearing in equation (71) and the functions tabulated in table VI is:

$$\begin{aligned} \alpha_{\mathrm{N}} &= \alpha_{\mathrm{Nl}} + \mathrm{M}_{\mathrm{r}}^{2} \alpha_{\mathrm{N2}} + \mathrm{K} \alpha_{\mathrm{N3}} \\ \beta_{\mathrm{N}} &= \beta_{\mathrm{Nl}} + \mathrm{M}_{\mathrm{r}}^{2} \beta_{\mathrm{N2}} \\ B_{\mathrm{N}} &= B_{\mathrm{Nl}} + \mathrm{M}_{\mathrm{r}}^{2} B_{\mathrm{N2}} + \mathrm{K} B_{\mathrm{N3}} + \frac{\mathrm{K}}{\mathrm{M}_{\mathrm{r}}^{2}} B_{\mathrm{N4}} + \frac{\mathrm{K}^{2}}{\mathrm{M}_{\mathrm{r}}^{2}} B_{\mathrm{N5}} \end{aligned}$$

APPLICATION OF ANALYSIS

Before the results of the previous section may be applied it is necessary to determine the quantity ε , the coefficients a_N , and the reference conditions. The quantities ε and a_N , which represent the

magnitude and form, respectively, of the external velocity distribution, are determined from potential-flow theory or from experimental measurements. Because the results of this report apply primarily to the flow over thin two-dimensional wings at Mach numbers greater than 1, ε and a_N as obtained by linearized theory (ref. 16) will be presented herein. It is assumed that the coordinates of a wing section are known and can be fitted by a polynomial of fourth or lesser degree:

$$Y = b_1 x + b_2 x^2 + b_3 x^3 + b_4 x^4$$
(72)

The values of ε , a_N , and u_r are obtained by matching the expression for the velocity distribution obtained from linearized theory with equation (1):

$$\frac{u_e}{u_r} = 1 + \epsilon (a_1 x + a_2 x^2 + a_3 x^3)$$
 (1)

where

$$u_{r} = \left(1 - \frac{a_{1}}{\sqrt{M_{\infty}^{2} - 1}}\right) u_{\infty}$$

$$\varepsilon = \frac{-2b_{2}}{\sqrt{M_{\infty}^{2} - 1}}$$

$$3b_{3} \qquad 2b_{4}$$
(73)

$$a_1 = 1$$
 $a_2 = 2b_2$ $a_3 = b_2$
If the velocity distribution over the wing were known experimentally, the starting point of the calculation would be equation (1), with u_r , ε ,

starting point of the calculation would be equation (1), with u_r , ε , and a_N determined by fitting a polynomial to the measured velocities. If, in a particular application, a_2 or a_3 is much smaller than 1, then that term need not be included in the solution.

The reference Mach number and temperature are obtained from equations (10) and (73):

$$M_{r}^{2} = \frac{M_{\infty}^{2} \left(\frac{u_{r}}{u_{\infty}}\right)^{2}}{1 + \frac{\gamma - 1}{2} M_{\infty}^{2} \left[1 - \left(\frac{u_{r}}{u_{\infty}}\right)^{2}\right]}$$
(74)

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$$t_{r} = t_{\infty} \sqrt{\frac{1 + \frac{\gamma - 1}{2} M_{\infty}^{2}}{1 + \frac{\gamma - 1}{2} M_{r}^{2}}}$$
(75)

The results of the analysis will now be summarized as they are needed in a particular application. In the following equations the functions $g_n(\eta)$, $h_n(\eta)$, and $H_n(\eta)$ frequently appear. They are related to the tabulated functions in the following manner:

$$g_{N}(\eta) = g_{NL}(\eta) + M_{r}^{2} g_{N2}(\eta) + Kg_{N3}(\eta)$$
(40)

$$h_{N}(\eta) = h_{Nl}(\eta) + M_{r}^{2} h_{N2}(\eta)$$
(44)

$$H_{N}(\eta) = H_{N1}(\eta) + M_{r}^{2} H_{N2}(\eta) + KH_{N3}(\eta) + \frac{K}{M_{r}^{2}} H_{N4}(\eta) + \frac{K^{2}}{M_{r}^{2}} H_{N5}(\eta)$$
(47)

The constant K is related to the given wall temperature for flows with arbitrary rates of heat transfer:

$$K = \frac{1}{s(0)} \left\{ \frac{t_{w}}{t_{r}} - 1 - \frac{\gamma - 1}{2} M_{r}^{2} r(0) \right\}$$
(34)

where s(0) and r(0) appear in table I. (For flows with zero heat transfer, K = 0.)

The velocity profile is given by

$$\frac{u}{u_{r}} = \frac{1}{2} f'(\eta) + \varepsilon a_{N} x^{N} g'_{N}(\eta)$$
(51)

where the repeated index N indicates a summation over all values of N. The temperature profile, for zero heat transfer, is

$$\frac{\mathbf{t}}{\mathbf{t}_{\mathbf{r}}} = \mathbf{1} + \frac{\gamma - \mathbf{1}}{2} \, \mathbf{M}_{\mathbf{r}}^{2} \left[\mathbf{r}(\eta) - 2\boldsymbol{\epsilon} \, \mathbf{a}_{\mathbf{N}} \mathbf{x}^{\mathbf{N}} \, \mathbf{h}_{\mathbf{N}}(\eta) \right]$$
(52)

and for flows with heat transfer, is

$$\frac{t}{t_r} = 1 + K_B(\eta) + \frac{\gamma - 1}{2} M_r^2 \left[r(\eta) - 2\epsilon a_N x^N H_N(\eta) \right]$$
(53)

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These profiles can be obtained in terms of the physical variable y by the following relations between η and y: (a) for zero heat transfer,

$$y = 2 \sqrt{\frac{\nu_{r} xC}{u_{r}}} \left\{ \left[1 + \frac{\gamma}{2} M_{r}^{2} \epsilon a_{N} x^{N} \right] \left[\eta + \frac{\gamma - 1}{2} M_{r}^{2} Ir(\eta) \right] - (\gamma - 1) M_{r}^{2} \epsilon a_{N} x^{N} Ih_{N}(\eta) \right\}$$

$$(55)$$

(b) for arbitrary rates of heat transfer,

$$y = 2 \sqrt{\frac{\nu_{r} xC}{u_{r}}} \left\{ \left[1 + \frac{\gamma}{2} M_{r}^{2} \varepsilon a_{N} x^{N} \right] \left[\eta + \frac{\gamma - 1}{2} M_{r}^{2} Ir(\eta) + KIs(\eta) \right] - (\gamma - 1) M_{r}^{2} \varepsilon a_{N} x^{N} IH_{N}(\eta) \right\}$$
(56)

The functions f, r, and s appear in table II. The functions g appear in table III; h, in table IV; and H, in table V.

The constant C is defined by

$$C = \sqrt{\frac{t_{w}}{t_{r}}} \frac{(t_{r} + 216^{\circ} R)}{(t_{w} + 216^{\circ} R)}$$
(3)

where t_w is given for flows with heat transfer, while for zero heat transfer a mean value of the adiabatic wall temperature is used. The adiabatic wall temperature is

$$t_{aw} = t_r \left\{ 1 + \frac{\gamma - 1}{2} M_r^2 \left[r(0) - 2\varepsilon a_N x^N h_N(0) \right] \right\}$$
(65)

The temperature recovery factor is

$$F_{\rm R} = r(0) - 2\epsilon a_{\rm N} x^{\rm N} M_{\rm r}^2 h_{\rm N2}(0)$$
 (67)

The following results were found for local and average skin-friction coefficients and for a heat-transfer parameter:

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$$\mathbf{e}_{\mathbf{f}} = \frac{1}{2} \sqrt{\frac{\mathbf{C}}{\mathrm{Re}}} \left\{ \mathbf{f}^{"}(\mathbf{0}) + 2\varepsilon \ \mathbf{a}_{\mathbf{N}} \mathbf{x}^{\mathbf{N}} \left[\mathbf{g}_{\mathbf{N}}^{"}(\mathbf{0}) - \frac{\gamma}{4} \ \mathbf{M}_{\mathbf{r}}^{2} \ \mathbf{f}^{"}(\mathbf{0}) \right] \right\}$$
(61)

$$c_{\rm F} = \sqrt{\frac{c}{\rm Re}} \left\{ f''(0) + \frac{2\epsilon \ a_{\rm N} x^{\rm N}}{2{\rm N}+1} \left[g_{\rm N}''(0) - \frac{\gamma}{4} \ M_{\rm r}^2 \ f''(0) \right] \right\}$$
(62)

and

$$Nu = \frac{\sqrt{C Re}}{2(t_{aw}^{*} - t_{w}^{*})} \left\{ Ks'(0) - (\gamma - 1) M_{r}^{2} \varepsilon a_{N} x^{N} \left[H_{N}'(0) + \frac{\gamma}{2(\gamma - 1)} Ks'(0) \right] \right\}$$
(64)

where

$$Re \equiv \frac{u_r x}{v_r}$$
$$Nu \equiv \frac{C t_w^* x}{t_{aw}^* - t_w^*} t_y^* \big)_w$$

All initial values [f"(0), etc.] are tabulated in table I. The displacement thickness for flows with arbitrary heat-transfer rates is given by

$$\delta^{*} = \sqrt{\frac{\nu_{r} \mathbf{x}^{\mathrm{C}}}{u_{r}}} \left\{ \left[1 + \frac{\gamma}{2} \mathbf{M}_{r}^{2} \varepsilon \mathbf{a}_{\mathrm{N}} \mathbf{x}^{\mathrm{N}} \right] \left[1.7208 + 4.0218 \mathbf{K} + (\gamma - 1)(1.1094 \mathbf{M}_{r}^{2}) \right] + \varepsilon \mathbf{a}_{\mathrm{N}} \mathbf{x}^{\mathrm{N}} \left[\alpha_{\mathrm{N}} + (\gamma - 1) \mathbf{M}_{r}^{2} \mathbf{B}_{\mathrm{N}} \right] \right\}$$
(71)

For flows with zero heat transfer, K will vanish and B_N is replaced by β_N in the last equation. (Values of α_N , β_N , and B_N can be found in table VI.)

The results of this analysis are not necessarily limited to the integral values of N for which calculations were made. Interpolation of the results presented in the tables will yield valid results for other values of N. Values for N = 0 are included in order to facilitate this interpolation (see appendix C and table I).

The equations presented in this section apply also for flat-plate flows. For this special case, $\varepsilon = 0$ and the reference conditions are equal to the undisturbed free-stream conditions.

DISCUSSION OF EXAMPLES

The results of the previous section were applied to two representative wings in order to determine the combined effects of heat transfer and pressure gradient on boundary-layer characteristics. Cross-sectional views of the forward portion of these wings are shown in figure 4. The first of the two wings has a constant adverse pressure gradient, while the second has a constant favorable pressure gradient. A maximum thickness ratio of 0.05 and a free-stream Mach number of 3 were chosen for the wing segments of both examples. The velocity and temperature distributions at the outer edge of the boundary layer are shown in figure 5.

The local skin-friction parameter $c_f \sqrt{Re/C}$ for both representative wings, computed by the present method, is presented in figure 6. The effect of pressure gradient in the absence of heat transfer $\left[\frac{\partial t}{\partial y}\right]_{w} = 0$

is to decrease skin friction for flows with adverse pressure gradients, and to increase skin friction for flows with favorable gradients. (For flows with zero pressure gradients, $c_{f}\sqrt{Re/C} = 0.664$ for all values of x as indicated by a dashed line in the figure.) The effects of pressure gradient are accentuated by adding heat to the boundary layer. For the present examples, the aforementioned decrease and increase in skin friction is doubled when the wall is heated to approximately four times the ambient air temperature. Sufficient cooling at the wall, on the other hand, appears to reverse the trend of the pressure gradient alone. Thus, for a wall temperature approximately equal to one-fourth the ambient air temperature, there is a slight increase in skin friction for flows with adverse pressure gradients; whereas there is a decrease in the case of favorable pressure gradients. The average friction drag parameter $C_{\rm F} \sqrt{{\rm Re}/{\rm C}}$, as shown in figure 7, exhibits the same trends as the local skin friction.

The friction-drag parameter $C_{\rm F}\sqrt{{\rm Re}/{\rm C}}$ is useful because it applies at all flight altitudes, and the actual velocity, density, and temperature need not be specified a priori. On the other hand, it is a misleading parameter, because the viscosity-temperature dependence factor C, which is a function of the wall temperature ratio, is affected by the rate of heat transfer at the wall. For this reason, the average friction drag coefficient multiplied by $\sqrt{{\rm Re}}$ was found for the two representative wings at conditions existing at 35,000 feet, as shown in figure 8. The rate of change of friction drag along the surface is nearly the same as was shown in figure 7. The relative magnitudes of the friction drag curves are altered, however, so that the highest drag is found for the cooling case, while the lowest drag is obtained with the hot wall, regardless of the type of pressure gradient.

The heat-transfer parameter $Nu/\sqrt{Re C}$ for the two representative wings is plotted in figure 9. The local rate of heat transfer was found to increase along the wing when the pressure gradient and wall temperature were such that the skin friction decreased and vice versa.

The temperature recovery factor, as plotted in figure 10, was found to vary slightly as a result of the pressure gradient. The variation is

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of the same order of magnitude as the pressure gradient, and hence a much larger change might be expected for larger pressure gradients. On the other hand, the variable term in the expression for the recovery factor (eq. (67)) is proportional to the square of the Mach number and would be unimportant at low speeds. It is therefore not surprising that recovery factors obtained by the present method do not agree with those obtained in reference 17, where the variation of fluid properties was neglected.

Velocity profiles at the midchord point (x = 1) of the two wings are presented in figure 11. The effect of heat transfer on the local velocity in the boundary layer is seen to be quite large - there is a marked thinning of the boundary layer when heat is extracted and a thickening when heat is added. Although the local velocity and its first derivative are altered only slightly because of the pressure gradient, the local curvature of the profiles appears to be affected to a greater extent. In particular, the profiles for zero heat transfer and a hot wall have an inflection point when the pressure gradient is adverse; whereas no inflection point is evident when the pressure gradient is favorable, even when the wall temperature is four times the ambient air temperature. In general, a velocity profile without an inflection point. (The shape of the temperature profile, however, also affects the criterion of stability.)

Although the local velocity near the outer edge of the boundary layer did not exceed the free-stream velocity, as discussed in reference 18, the functions $g'(\eta)$ are of a form indicating that such an overshoot may exist for slightly larger pressure gradients. (See fig. 1.)

Temperature profiles for the example wings are plotted in figure 12. These profiles do not differ greatly with the two different pressure gradients. The effect of heat transfer is quite large, however, as is evident from a comparison of the extremely thin profiles when the wall temperature ratio is 0.25 with the relatively thick profiles when this ratio is 4.

The ratio of the displacement thickness along the example wings δ^* to the displacement thickness along an equivalent flat plate δ^*_{FP} is plotted in figure 13. The displacement thickness is found to be less than the flat plate value for the adverse pressure gradient and greater for the favorable pressure gradient. This behavior is opposite to the trend found for incompressible flow and can be explained as follows: The ratio of displacement thicknesses is found to be

$$\frac{\delta^{*}}{\delta_{\mathrm{FP}}^{*}} = 1 + \frac{\gamma}{2} M_{\mathrm{r}}^{2} \varepsilon a_{\mathrm{N}} x^{\mathrm{N}} + \frac{\varepsilon a_{\mathrm{N}} x^{\mathrm{N}} \left[\alpha_{\mathrm{N}} + (\gamma - 1) M_{\mathrm{r}}^{2} B_{\mathrm{N}} \right]}{1.72 + 4.02 \mathrm{K} + (\gamma - 1)(1.11) M_{\mathrm{r}}^{2}}$$
(76)

For incompressible flow and zero heat transfer, equation (76) reduces to

$$\frac{\delta^{*}}{\delta^{*}_{FP}} = 1 - 2.6 \varepsilon a_{N} x^{N}$$
(77)

In a favorable gradient ($\varepsilon a_N x^N$ positive), δ^*/δ_{FP}^* as expressed by equation (77) decreases; whereas the ratio increases in an adverse gradient. This well-known thinning or thickening of the boundary layer is essentially an effect of the change in local Reynolds number caused by the change in the external velocity.

As the Mach number is increased, however, the term $\frac{\gamma}{2} M_r^2 \varepsilon a_N x^N$ in equation (76) becomes of importance. This term is related to the change in density at the outer edge of the boundary layer. Its significance may be qualitatively determined by supposing for the moment that viscosity may be neglected and by consideration of the twodimensional compressible vorticity transport equation for an inviscid fluid

$$\frac{D}{Dt}\left(\frac{\Omega}{\rho}\right) = 0 \tag{78}$$

This expression shows that the vorticity changes in the same sense as the density. In a favorable pressure gradient, therefore, the vorticity will decrease along the wing because the density decreases along the wing. A decrease of vorticity in the boundary layer tends to thicken this layer.

If the complete equation for a viscous fluid is considered, there may be two opposing effects which occur at high Mach numbers: The effect of a favorable pressure gradient on Reynolds number (and hence viscosity) tends to thin the boundary layer; at the same time, the effect of the favorable pressure gradient on the vorticity directly tends to thicken the boundary layer. (A similar argument applies to adverse pressure gradients.) At a sufficiently high Mach number this second effect will predominate, as was found in the case of the present examples. For the case of constant pressure gradients and zero heat transfer, it can be shown that the aforementioned reversal of trends in the function $\delta^*/\delta^*_{\rm FP}$ occurs at a Mach number of 1.76.

If the Mach number is further increased, the thickening or thinning of the boundary layer will also affect the slope of the velocity profiles at the wall and hence the skin friction. For small constant pressure gradients and zero heat transfer, the skin friction trends are found to reverse at a Mach number of 4.71.

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The effect of Prandtl number on the local friction drag parameter over the wing with the adverse pressure gradient is shown in figure 14. At midchord, a Prandtl number of 1 yields a friction drag coefficient 4 percent lower than a Prandtl number of 0.72 when the wall is insulated. A solution for a Prandtl number of 1 for flows with heat transfer was not obtained, but it is expected that the effect would be considerably larger than the 4 percent found for flows with zero heat transfer.

As a check on the accuracy of the present method, the solution for Prandtl number 1 and zero heat transfer was compared with an exact solution of Howarth (refs. 13 and 14), which applies even at pressure gradients as large as required for separation. At the midchord station the local friction drag parameter agrees within 0.7 percent with that obtained by Howarth.

CONCLUDING REMARKS

A method for the calculation of compressible laminar boundary layer characteristics for flows with heat transfer and small arbitrary pressure gradients is presented. This method was applied to the flow over two representative wings - one with a constant adverse pressure gradient, the other with a constant favorable pressure gradient. The investigation led to the following conclusions: It was found that the deviations in skin friction caused by the pressure gradient were magnified when the wall was heated and reduced when the wall was cooled. Large amounts of cooling were found to reverse the rate of change of skin friction along the wing caused by a pressure gradient alone.

Local rates of heat transfer were found to vary in direct opposition to the skin friction: If the pressure gradient was such that the shearing stress decreased along the wing, then the heat-transfer rate increased, and vice versa.

Temperature recovery factors were found to be affected by the pressure gradient. The percentage change in recovery factor along the wing was somewhat smaller than the percentage change in the external velocity.

The displacement thickness at a Mach number of 3 was found to be greater than the displacement thickness of an equivalent flat plate when the pressure gradient is favorable and less than the flat plate displacement thickness for the adverse pressure gradient. This result is opposite to the trend found at low speeds.

Lewis Flight Propulsion Laboratory National Advisory Committee for Aeronautics Cleveland, Ohio, August 19, 1953

APPENDIX A

SYMBOLS

The following symbols are used in this report: A1, A2, ... arbitrary constants (eq. (C2)) a measure of the shape of the external velocity distribution a_N function appearing in equation (71) B_{N} b₁,b₂,... arbitrary constants (eq. (76)) C constant of proportionality in viscosity-temperature relation average friction drag coefficient = $\frac{1}{\frac{1}{2}\rho_r u_r^2 x} \int_0^r \tau_w dx$ C_F local friction drag coefficient = $\frac{\tau_w}{\frac{1}{2}\rho_r u_r^2}$ °f specific heat at constant pressure c_p $\mathbf{F}_{\mathbf{R}}$ temperature recovery factor f solution of zero-order momentum equation function defined in equations (C4) and (C7) G solution of first-order momentum equation g Η solution of first-order energy equation with heat transfer h solution of first-order energy equation without heat transfer factor describing heat-transfer conditions (eq. (34)) ĸ k thermal conductivity Μ Mach number exponent in free-stream velocity distribution, $(u_e^* = 1 + \epsilon a_N x^N)$ Nusselt number = $\frac{C t_w^* x}{t_{aw}^* - t_w^*} \frac{\partial t^*}{\partial y} \Big)_w$ N Nu transformed variable n Prandtl number = $\mu c_{p}/k$ \Pr

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р	static pressure
đ	local rate of heat transfer
R	gas constant
Re	Reynolds number = $u_r x / v_r$
r	solution of zero-order energy equation
S	Sutherland's constant
S	solution of zero-order energy equation
Т	total temperature
t	static temperature
u	velocity in x-direction
v	velocity in y-direction
x	distance along surface measured from leading edge
Y	normal coordinate of surface
У	distance from surface measured perpendicular to surface
$\left. \begin{smallmatrix} \alpha_{\mathbb{N}} \\ \beta_{\mathbb{N}} \end{smallmatrix} \right\}$	functions appearing in equation (71)
r	ratio of specific heats
δ*	displacement thickness
3	small quantity - a measure of magnitude of velocity dis- tribution at edge of boundary layer
η	characteristic variable defined by equation (26)
θ	dummy variable
μ	coefficient of viscosity
ν	kinematic viscosity = μ/ρ
ξ	dummy variable

 τ shearing stress

 φ transformed stream function

ψ stream function

$$vorticity, \left(\frac{\partial v}{\partial x} - \frac{\partial u}{\partial y}\right)$$

Subscripts:

aw	adiabatic wall
e	conditions at outer edge of boundary layer
FP	equivalent flat-plate value
r	reference condition
W	conditions at wall or surface
æ	undisturbed free-stream condition
x,y,n	partial differentiation with respect to x, y, or n
М	value of function corresponding to given value of M
N	value of function corresponding to given value of N

Superscripts:

differentiation with respect to η

Special Notation

A bar over a quantity indicates the order of approximation. (A single bar signifies a zero-order quantity, double bar signifies a first-order quantity, etc.)

A repeated index N appearing on a and one or more other symbols indicates summation: $\begin{bmatrix} 1 + \varepsilon & a_N x^N \equiv 1 + \varepsilon (a_1 x + a_2 x^2 + \dots) \end{bmatrix}$. The symbol I preceding a quantity indicates integration from zero to η : for example, $Ir(\eta) = \int_0^{\eta} r(\xi) d\xi$.

APPENDIX B

NUMERICAL SOLUTION OF DIFFERENTIAL EQUATIONS

By Lynn U. Albers

Each of the ordinary differential equations for f, r, s, g, h, and H with its associated boundary conditions at zero and infinity constitutes a two-point boundary value problem. With the exception of the Blasius equation all equations are linear, and the principle of superpositions of any two solutions may be used to satisfy the boundary conditions at infinity. Usually, two solutions close to the correct one were used in the final combination in order to minimize round-off errors. All integrations were performed on the IBM Card-Programmed Electronic Calculator. The combination of solutions and rounding to four decimal places was accomplished on the IBM Type 604 Calculating Punch by using general purpose floating-point control panels.

The integration technique will be described for the g problem, but it will be applicable to all the other problems with slight modifications. If $g''(\eta)$ is given at five values of η , a fourth-degree polynomial in η may be passed through the set of values; and if g, g', and g'' are known at the fifth point, the polynomial representation of g''' may be integrated to yield g, g', and g'' at the next (sixth) point. These quantities may then be substituted in the differential equation (41) to yield g'' at the sixth point. Thus, by using the five previous points, the integration may be extended one step at a time.

The integration was initiated with an assumed trial value of g''(0)and a value of g'''(0) calculated from the equation. This value of g'''(0) was also used as a first estimate of g''' at the next four points. The fourth-degree polynomial representing g''' over this range was then integrated to yield g, g', and g'' at the second point. Substitution in the equation then yielded a better estimate of g''' at the second point. Integration of the fourth-degree polynomial representation of g'''' from zero to successive points was alternated with substitution in the equation to improve values of g''' in an iterative fashion until convergence was obtained at the five initial points.

It was found that when g' was close to its boundary value at infinity, the regular integration process encountered oscillations in the function g''. To avoid this phenomenon, a procedure analogous to the starting procedure was used in an iterative manner. This smoothing process was used from $\eta = 3.4$ on. Integration was carried to a point which would yield four-decimal-point accuracy in the value of g"(0) and in the g' and g" data.

All integrations were performed using a step size of 0.1. Subsequent investigation of the effect of step size indicated that tabular values of the functions f, r, and s are correct as presented in table II, while the functions g, h, and H may be in error by 1 in the fourth place.

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APPENDIX C

SOLUTION OF FIRST ORDER-EQUATIONS FOR N = 0

Physically, the solution of the first-order equations for N = 0is of little interest because the external flow represented by $u_e^{\#} = 1 + \varepsilon$ is simply the flow over a flat plate; the term ε arises because the reference velocity is taken slightly different from the stream velocity. In practical applications, flat-plate flows would be handled by the zero-order solutions. The case of N = 0 may be of academic interest, however, in addition to supplying limiting conditions for cases of $N \neq 0$.

The first-order momentum equation (eq. (38)) for N = 0 becomes

$$g_{01}^{""} + f g_{01}^{"} + f^{"}g_{01} = 0$$
(C1)
$$g_{01}(0) = g_{01}^{i}(0) = 0$$
$$g_{01}^{i}(\infty) = 1$$

The functions g_{02} and g_{03} vanish identically. The general solution of this equation is

$$g_{0}(\eta) = A_{1}f' + A_{2}(f + f'\eta) + A_{3}\left[(f + f'\eta) \int_{0}^{\eta} \frac{f'f'' d\eta}{(2f'^{2} - ff'')^{2}} - f'' \int_{0}^{\eta} \frac{(f + f'\eta)f'' d\eta}{(2f'^{2} - ff'')^{2}}\right]$$
(C2)

The coefficient of A_2 in equation (C2) was given in reference 19. From the boundary conditions it can be found that $A_1 = A_3 = 0$ and $A_2 = \frac{1}{4}$. Therefore,

 $g_{Ol}(\eta) = \frac{1}{4} (f + f'\eta)$ (C3)

The first-order energy equation for N = 0 and zero heat transfer reduces to

$$h_{Ol}^{"} + \Pr f h_{Ol}^{'} = \Pr (g_{Ol}r' + f'' g_{Ol}^{"}) = G_{l}(\eta)$$
(C4)

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$$h_{Ol}(0) = 0$$
 $h_{Ol}(\infty) = 1$

Equation (C4) is a first-order linear equation in h_0^i . The solution satisfying the boundary conditions is

$$h_{Ol}(\eta) = l - \int_{\eta}^{\eta} \left[f''(\xi) \right]^{Pr} \int_{0}^{\xi} \frac{G_{l}(\theta)}{\left[f''(\theta) \right]^{Pr}} d\theta d\xi \qquad (C5)$$

The function h_{O2} is identically equal to zero. For flows with arbitrary rates of heat transfer, the following equations arise for N = 0:

$$H_{Ol}'' + \Pr f H_{Ol}' = \Pr \left[g_{O}r' + f'' g_{O}''\right] = G_{l}(\eta)$$
(C6)

$$H_{04}'' + \Pr f H_{04}' = \frac{2 \Pr}{\gamma - 1} g_0 s' = G_4(\eta)$$
 (C7)

$$H_{0}(0) = 0 \qquad H_{01}(\infty) = 1 \qquad H_{04}(\infty) = 0$$

$$H_{02} = H_{03} = H_{05} \equiv 0$$

The solution of equation (C6) satisfying the appropriate boundary conditions is

$$H_{01}(\eta) = 1 - \int_{\eta}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{1}(\theta) d\theta d\xi - \int_{0}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{1}(\theta) d\theta d\xi \int_{\eta}^{\infty} [f''(\xi)]^{Pr} d\xi$$

$$\int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi \int_{\eta}^{\infty} (C8)^{Pr} d\xi \int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi \int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi \int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi$$

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Similarly, the solution of equation (C7) is

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$$H_{04}(\eta) = - \int_{\eta}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{4}(\theta) d\theta d\xi + \frac{\int_{0}^{\infty} [f''(\xi)]^{Pr} \int_{0}^{\xi} [f''(\theta)]^{-Pr} G_{4}(\theta) d\theta d\xi}{\int_{0}^{\infty} [f''(\xi)]^{Pr} d\xi} \int_{\eta}^{\infty} [f''(\xi)]^{Pr} d\xi$$
(C9)

The function $H_0(\eta)$ is again obtained by a linear combination of $H_{Ol}(\eta)$ and $H_{O4}(\eta)$ in the following manner:

$$H_{O}(\eta) = H_{Ol}(\eta) + \frac{K}{M_{r}^{2}} H_{O4}(\eta)$$
(Clo)

Values of $h_0(0)$ and $H_0'(0)$ for Pr = 0.72 were obtained by numerical integration and are listed in table I.

APPENDIX D

SOLUTION FOR
$$Pr = 1$$

In order to establish the effect of Prandtl number on skin friction and to provide a basis for comparison with other solutions, some of the energy equations were solved for the special case of Pr = 1. The solution of the zero-order energy equation for Pr = 1 is

$$r(\eta) = 1 - \frac{1}{4} (f')^2$$
 (D1)

$$s(\eta) = 2 - f'(\eta)$$
 (D2)

The solution of the first-order energy equations for zero heat transfer is, for Pr = 1,

$$\begin{array}{c} \mathbf{h}_{\mathrm{NL}}(\eta) = \frac{1}{2} \mathbf{f}^{\dagger} \mathbf{g}_{\mathrm{NL}}^{\dagger} \\ \\ \mathbf{h}_{\mathrm{N2}}(\eta) = \frac{1}{2} \mathbf{f}^{\dagger} \mathbf{g}_{\mathrm{N2}}^{\dagger} \end{array} \right\}$$
(D3)

and

The linear combination of equations (D3) yields

$$h_{N}(\eta) = \frac{1}{2} f' \left[g_{N1}' + M_{r}^{2} g_{N2}' \right]$$
 (D4)

The function g_{Nl} is independent of Prandtl number, and hence the values appearing in table III apply for all Prandtl numbers. The function g_{12} was calculated numerically for a Prandtl number of 1, and results of this calculation appear in table VII.

The complete solution of the first-order energy equation for Pr = 1 and flows with arbitrary rates of heat transfer was not found. The following are the solutions of equations (45) and (46) for Pr = 1 and heat transfer:

$$\begin{array}{c} H_{N1}(\eta) = \frac{1}{2} f' g_{N1}' \\ H_{N2}(\eta) = \frac{1}{2} f' g_{N2}' \end{array} \right\}$$
(D5)

and for Pr = 1 and N = 0:

$$H_{04}(\eta) = \frac{\eta f''(\eta)}{2(\gamma-1)}$$
(D6)

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TABLE I. - INITIAL VALUES

[Pr, 0.72; r, 1.4]

$$f''(0) = 1.3282$$
 $r(0) = 0.8477$

s(0) = 2.0748

s'(0) = -1.2267

	N = 0	N = 1.	N = 2	$\mathbb{N}=3$
g"(0)	0.9962	4.0821	6.3546	8.2879
g <mark>"</mark> (0)	0	.2807	.5847	.8717
gn(0)	о	5.0447	8.9738	12.4065
h _{N1} (0)	0.1523	0.1524	0.1526	0.1528
h _{N2} (0)	0	.0085	.0123	.0146
H'N1 (0)	0.0904	0.1479	0.1802	0.2042
н _{N2} (0)	σ	.0082	.0145	.0195
H'(0)	о	4201	2200	.0899
H:(0)	1.5326	5,5574	8.6985	11.3879
н <u>;</u> (0)	0	5.3452	10.1820	14.5535
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				(Pr	, 0.72]	9	NACA	•	
n	ŕ	L,	f"	Ir	r	г	Is	6	B [†]
0 .1 .2 .3 .4	0,0000 .0066 .0266 .0597 .1061	0.0000 1326 2655 3979 25294	1.3282 1.5279 1.5259 1.3269 1.3205 1.3095	0.0000 .0047 1657 2514 .3323	0.8477 .8445 .8350 .8194 .797%	0.0000 0635 1360 1893 3503	0.0000 .2013 .3904 .5672 .7319	2.0740 1.9531 1.8295 1.7071 1.5853	-1.3367 -1.3365 -1.3352 -1.3315 -1.3143
5.67 .2 .2 .2	1656 3379 3230 4203 5295	6596 7875 9123 10335 11495	1,8980 1,2663 1,8314 1,1866 1,1317	.4107 .4860 .5576 .6252 .6862	.7392 .7366 .6969 .6536 .6536	3056 3626 4110 4522 4046	.8843 1.0346 1.1534 1.8704 1.3761	1.4644 1.3450 1.3276 1.1139 1.0016	-12020 -1.1853 -1.1617 -1.1311 -1.0933
1.0 1.1 1.8 1.5 1.4	.6500 .7012 .9223 1.6725 1.8310	1.2 5951.36261.45791.54491.6230	1.0670 .9994 .9184 .8259 .7361	.7404 .7998 .8476 ルシの3 .シミしの	.5570 ,5057 .4536 .4030 .3519	5071 6189 5198 5101 4906	1.4709 1.5552 1.6295 1.0945 1.7508	.8945 .7983 .6957 .6052 .5215	-10478 -9958 -9958 -9361 -8713 -8020
1.b 1.6 1.7 1.0 1.5	1.3968 1.5691 1.7469 1.9295 21160	1,6921 1,7522 1,6035 1,8467 1,8382	.6455 .5500 .4715 .J954 .J205	9608 9090 1.0120 1.0326 1.0494	.3042 .3590 .8187 .1019 .1094	4627 4282 3889 3470 3042	1.7990 1.8400 1.8744 1.9030 1.9265	.4449 .3757 .3138 .8592 .8117	-7296 -550 -5820 -5820 -3099 -4406
2.0 2.1 2.8 2.3 2.4	8.3057 7.4930 2.6524 2.6582 2.6582 5.0553	19110 19339 19517 19684 19756	2569 2031 1509 2179 2075	1.0689 1.0737 1.0384 1.0891 1.0944	.1211 .0969 .0765 .0597 .0459	2628 8228 1854 1588 1831	19456 19609 19730 19826 19899	.1709 .1364 .1075 .0837 .0644	3759 3168 8683 8146 1730
845 857 857 84 1 1 1	3.8 8 3 3 3.4 8 1 9 3.6 8 6 9 3.8 8 0 3 4.0 7 9 9	1,9031 1,9885 1,9985 1,9950 1,9957	.0536 .0454 .0317 .0217 .0146	$1.09 & 4 \\ 1.1 & 0.1 & 4 \\ 1.1 & 0.3 & 7 \\ 1.1 & 0 & 5 & 4 \\ 1.1 & 0 & 5 & 6 \\ 1.1 & 0 & 5 & 6 \end{bmatrix}$.0349 .0862 .0194 .0143 .0102	0951 0770 0595 0454 0341	1,9956 1,9998 2,0030 2,0053 2,0070	.0489 .0366 .0271 .0198 .0143	1376 1078 0833 0635 0477
3.0 3.1 3.2 3.3 3.4	43796 14794 46793 48793 50793	1.9975 1.9957 1.9957 1.993 1.9945 1.9945 1.5997	.0026 .0057 .0059 .0088 .0088	1.1075 1.1081 1.1085 1.1065 1.1060 1.1090	.0077 .0051 .0035 .0027 .0016	0252 0104 0132 0094 0063	20083 20091 20097 20197 20101 20104	.0102 .0072 .0050 .0034 .0023	0353 0257 0185 0131 0093
34 34 34 34 34	5.8798 5.4798 5.6798 5.5798 6.0793	1.5590 1.4595 30000 20000 20000 20000	.0009 .0005 .0005 .0007 .0007	1,1 091 1,1 092 1,1 095 1,1 095 1,1 095	.0011 .0007 .0004 .000- .000-	0045 0031 0030 0013 0009	20100 20107 20103 20108 20108 20108	$\begin{array}{c} .0015\\ .0010\\ .0006\\ .0004\\ .0003\end{array}$	-2065 -2045 -2026 -2026 -2013 -2013
4.0 4.1 4.2 4.3 4.4	6.8798 6.4792 6.6792 6.3792 7.0792 7.0792	20000 2000 20000 20000 20000 20000 20000	0000 00000 00000 00000	11093 11093 11093 11093 1.1093 1.1094	.0001 .0001 .0000 .0000 .0000	00020002000200010001	8.0109 5.0109 8.0109 2.0109 2.0109 2.0109	.0002 .0001 .0001 .0000 .0000	- 0000 - 0005 - 0005 - ん003 - ん003
1	7.5798	3.0000	.0000	1.1094	.0000	0001	-010Y	.0000	01

TABLE II. - SOLUTIONS OF ZERO-ORDER MOMENTUM AND EMERGY EQUATIONS

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TABLE III. - SOLUTIONS OF FIRST-ORDER NOMENTUM EQUATION

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[Pr, 0.72; y, 1.40]

η	5 11	512	813	8 ₁₁	5 <u>12</u>	813	8 <mark>11</mark>	8 <mark>1</mark> 2	813	М
0 .1 .3 .3 .4	0.0000 .0197 .0763 .1657 .2839	0.0000 .0013 .0047 .0096 .0154	0.0000 .0839 .0902 .1913 .3803	0.0000 .3882 .7364 1.0446 1.3129	0.0000 .0247 .0427 .0543 .0599	0.0000 .4638 .8495 1.1620 1.4063	4.0 8 8 1 3.6 8 8 1 3.2 8 2 1 2.8 8 2 3 2.4 8 2 9	0.2807 .2133 .1474 .0851 .0379	5.0 4 4 7 4.2 3 9 4 3.4 8 3 1 2.7 7 5 8 2.1 1 7 6	1 1 1 1 1
-5 -6 .7 .8 .9	.4269 .5908 .7716 .9655 1.1685	.0 2 1 4 .0 2 7 2 .0 3 2 4 .0 3 6 6 .0 3 9 7	.4705 6358 .8106 .9900 1.1693	15413 17300 18795 19907 20648	.0601 .0557 .0475 .0365 .0237	1.5872 17098 17790 1.8003 17791	2,0849 1,6903 1,3018 .9238 .5619	0225 0647 0977 1207 1334	1.5089 .9505 .4438 0089 4058	1 1 1 1
1.0 1.1 1.3 1.3 1.4	1.3773 1.5882 1.7984 8.0050 2.3057	.0413 .0417 .0407 .0386 .0356	13446 15125 16703 18159 19478	31038 31104 30878 20403 19784	.0101 0032 0155 0261 0344	1.7213 1.6328 1.5200 1.3893 1.2470	.2226 -0862 3577 5651 7635	1360 1395 1152 0951 0712	7428 -1.0171 -1.2382 -1.3753 -1.4597	111111
1.5 1.6 1.7 1.8 1.9	2.3989 2.5833 2.7580 2.9228 3.0779	.0318 .0276 .0232 .0188 .0186	2.0651 2.1677 2.2556 2.3297 2.3911	1.8893 1.7961 1.6978 1.5990 1.5036	0403 0436 0446 0435 0408	1,0993 .9516 .8089 .6751 .5531	8904 9658 9988 9770 9858	0458 0311 .0018 .0198 .0338	-1.4856 -1.4591 -1.3884 -1.2832 -1.1540	1 1 1 1
2.0 2.1 2.3 8.3 2.4	3.2237 3.3611 3.4910 3.6143 3.7382	.0107 .0073 .0048 .0017 0003	2.4408 2.4805 2.5115 2.5353 2.5533	1,4147 1,3345 1,8643 1,2046 1,1551	0369 0323 0274 0226 0102	4448 3510 2719 2067 1542	8482 7533 5497 5450 4451	.0431 .0479 .0488 .0466 .0424	-10110 -8637 -7200 -5861 -4661	111111
2.5 2.6 2.7 2.8 2.9	38457 39856 40627 41677 42718	0019 0038 0041 0048 0055	25666 25768 25830 25878 25911	11153 10339 10598 10418 10286	0148 0108 0080 0058 0041	1189 0811 0578 0397 0870	-3543 -8750 -2083 -1541 -1113	•0368 •0308 •0249 •0195 •0148	- 3624 - 2755 - 2050 - 1493 - 1064	111111
3.0 3.1 3.2 3.3 3.3 3.4	4,3736 4,4751 4,5768 4,6768 4,7773	0056 0059 0060 0061 0063	25933 25948 25958 25964 25964 25968	1.0192 1.0126 1.0081 1.0051 1.0051	0028 0019 0013 0008 0005	Ω180 Ω118 Ω076 Ω048 Ω030	-0786 -0548 -0548 -0366 -0241 -0155	.0109 .0078 .0054 .0037 .0024	0744 0509 0342 0324 0144	11111
3.5 3.6 3.7 3.8 3.9	4,8775 4,9776 5,0777 5,1778 5,2778	0068 0063 0063 0063 0063	2.5970 2.5971 3.5978 2.5973 2.5973 2.5973	1.00191.00111.00071.00041.0002	0003 0003 0001 0001 .0001	0018 0011 0007 0004 0003	-0098 -0061 -0037 -0038 -0013	.0016 .0010 .0006 .0003 .0003	0092 0057 0035 0021 0012	111111
4.0 4.1 4.8 4.3 4.4	5.3778 5.4778 5.5778 5.6778 5.6778	0063 0063 0063 0063 0063	25973 25973 25973 25973 25973 25973	10001 10001 10000 10000 10000	.0000 .0000 .0000 .0000	0001 .0001 .0000 .0000 .0000	-0007 -0004 -0002 -0001 -0001	.0001 .0001 .0000 .0000 .0000	0007 0004 0003 0001 0001	11111
4.5	5.8778	0063	25973	1.0000	.0000	.0000	.0000	.0000	0000	1

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TABLE III Continued. 80	LUTIONS OF FIRST-ORDER	MOMENTUM EQUATION
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	$\{\Pr, 0.72; \gamma, 1.40\}$								MACA	
n	⁸ 21	5,22	⁶ 23	8'21	⁸ 22	^g 1 23	8 ["] 21	⁸ 22	5 ¹¹ 23	N
0 1935.4	0.0000 .0304 .1165 .2503 .4248	0.0000 .0027 .0099 .0203 .0327	0.0000 .0421 .1581 .3328 .5589	0.0000 .5955 11119 15512 19164	0.0000 .0517 .0901 .1159 .1301	0.0000 .8161 1.4772 1.9957 8.3848	63546 55873 47743 40167 32917	0.5847 4499 .3197 .1981 .0884	89738 75667 58766 45152 38878	8 N N N N
.5 .6 .7 .8 .9	.6318 .8648 1.1168 1.3830 1.6578	.0460 .0592 .0716 .0826 .0916	.8059 1.0810 1.3685 1.6601 1.9485	2.2108 2.4385 2.6037 2.7108 2.7647	.1341 .1893 .1175 .1005 .0801	2.6577 2.8280 2.9083 2.9109 2.8477	2,6041 1,9573 1,3538 .7964 .2886	0069 0858 1471 1903 2157	8.1948 1.2316 .3947 3819 9833	8 8 8 8 9 8 8 8 8 9 8 8 8 8 9 8 8 8 8 8
10 11 12 13 14	1,9343 2,2098 2,4798 2,7409 2,9906	.0985 .1032 .1057 .1062 .1062 .1050	2,2 2 7 8 2,4 9 3 0 2,7 4 0 4 2,9 6 7 0 3,1 7 1 1	2.7703 2.7335 2.6604 2.5575 2.4319	.0579 .0357 .0147 0039 0193	8,7299 2,5685 2,3738 2,1560 1,9247	-1654 -5604 -8914 -11542 -13461	2243 2181 1994 1712 1370	-1.4140 -1.7979 -2.0786 -2.2607 -2.3501	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
15 16 1.7 18 19	3.8268 3.4484 3.6549 3.8463 4.0832	.1025 .0989 .0947 .0902 .0857	3.3518 3.5090 3.6434 3.7563 3.8494	22907 21408 19887 18403 1.7002	0312 0393 0439 0454 0454	1.6888 1.4562 1.3337 1.0267 .8392	-1.4670 -1.5300 -1.5113 -1.4498 -1.3465	0999 0633 0296 0008 .0218	-2.3549 -8.2657 -8.1553 -1.9760 -1.7690	000000
8.0 8.1 2.2 2.3 2.4	4.1867 43381 44788 46105 4.7347	.0814 .0775 .0741 .0711 .0687	3.9248 3.9849 4.0317 4.0677 4.0947	1,5780 1,4581 1,3596 1,8767 1,8086	0413 0369 0319 0267 0216	6735 5307 4105 3117 2383	-1,8136 -1,0638 -9065 -,7589 -,6097	.0379 .0477 .0520 .0518 .0484	- 1.5 4 3 1 - 1.3 1 3 7 - 1.0 9 3 2 8 8 7 3 7 0 4 4	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
2.5 3.6 2.7 8.8 3.9	48587 49659 50754 51880 58866	.0668 .0653 .0648 .0633 .0633	4.1147 4.1398 4.1395 4.1467 4.1517	11542 11117 10793 10552 10376	017.1 0131 0098 0071 0051	1700 1821 0861 0596 0405	-4816 -3713 -3796 -2057 -1478	.0429 .0365 .0298 .0235 .0180	5468 4153 3087 3346 1600	00000
3.0 3.1 3.2 3.3 3.4	5,3 8 9 7 5,4 9 1 8 5,8 9 3 1 5,6 9 4 0 5,7 9 4 5	.0623 .0620 .0618 .0617 .0616	4.1550 4.1573 4.1587 4.1596 4.1608	1.0252 1.0165 1.0106 1.0065 1.0041	0035 0024 0016 0010 0006	0270 0177 0114 0072 0045	-1039 -0714 -0481 -0314 -0308	.0133 .0096 .0067 .0045 .0030	-1118 -0764 -0514 -0336 -0217	8 N N N N
3.8 3.6 3.7 3.8 3.9	5,8948 59950 6,0951 6,1952 6,2952	.0616 .0615 .0615 .0615 .0615 .0615	4.1605 4.1607 4.1609 4.1609 4.1610	1.00 % 5 1.00 1 5 1.00 0 8 1.00 0 5 1.00 0 3	0004 0008 0001 0001 .0001	0027 0016 0010 0006 0003	-0127 -0079 -0047 -0028 -0016	.0019 .0012 .0007 .0004 .0003	-0138 -0086 -0058 -0038 -0038 -0018	8 8 8 8 8 8 8 8 8 8
4.0 4.1 4.3 4.3 4.4	6,3953 6,4983 6,5953 6,6953 6,7983	.0615 .0615 .0615 .0615 .0615	4.1610 4.1610 4.1610 4.1610 4.1610 4.1610	1.0001 1.0001 1.0000 1.0000 1.0000	0000. 0000. 0000. 0000. 0000.	0008 0001 0000 0000 0000	0009 0005 0003 0001 0001	.0001 .0001 .0000 .0000 .0000	0011 0006 0004 0003 0001	8 N N N N
4.5	6.8983	.0615	4.1 61 0	10000	.0000	0000	0000	0000	.0000	8

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TABLE III: - Concluded. SOLUTIONS OF FIRST-ORDER MOMENTUM EQUATION

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	[Pr, 0.72; γ , 1.40]								NACA	
ή	g ₃₁	832	633	831	5 32	533	851	8 ₃₂	⁸ 33	N
0 1 8 7 4	0.0000 .0394 .1499 .3197 .5381	0.0000 0040 0147 0303 0488	0.0000 .0579 .9161 .4523 .7469	0.0000 .7690 1.4202 1.9587 8.3916	0.0000 .0770 1343 .1789 .1947	00000 11189 30067 36850 31772	8,2879 7,0947 5,9383 4,8448 3,8289	0.8717 .6698 .4766 .2986 .1409	13.4065 10.0003 7.7984 5.8139 4.0736	33533
.5 .6 .9	.7948 1.0806 1.3869 1.7063 8.0319	.0687 .0887 .1077 .1249 .1396	1.0823 1.4437 1.8181 2.1947 2.5646	2.7272 2.9739 3.1405 3.2353 3.8665	8018 1968 1881 1603 1339	3.5073 3.6982 3.7713 3.7460 3.6399	2.8970 2.0537 1.3933 .6163 .0198	.0064 1039 1865 8449 8794	2.5677 1.8856 .2079 6847 -1.4106	22222
1.0 1.1 1.9 1.3 1.4	8.3577 8.6786 8.9902 3.8890 3.5721	.1516 .1607 .1669 .1704 .1717	2.9805 3.2567 3.5687 .3.8536 4.1093	3.2 418 3.1 6 9 3 3.0 5 7 1 8.9 1 3 8 8.7 4 6 0	.1052 .0761 .0485 .0236 .025	3.4689 3.3474 2.9887 8.7052 8.4083	4988 9373 -1.2945 -1.5689 -1.7600	2922 2861 2645 3313 1906	-1.9855 -2.4220 -2.7306 -2.9202 -3.0003	33355
1.5 1.6 1.7 1.8 1.9	3.8377 4.0847 4.3186 4.5880 4.7137	.1711 .1690 .1659 .1681 .1581	4.3351 4.5318 4.6986 4.8390 4.9547	8-5639 2.3747 2.1858 2.0035 1.8330	0144 0268 0349 0392 0402	8.1084 1.8149 1.5355 1.8765 1.0425	-1.8696 -1.9088 -1.8658 -1.7719 -1.6315	1463 1021 0610 0254 .0033	-2.9815 -2.8767 -2.7009 -2.4711 -2.8051	33333
2.0 2.1 2.3 2.3 2.4	4.8892 50500 51981 53354 54638	.1548 .1504 .1471 .1442 .1418	5.0484 5.1229 5.1810 5.2256 5.2591	1.6782 1.5415 1.4239 1.3253 1.2448	0388 0356 0313 0265 0818	.8361 .6585 .5091 .3864 .2879	-1,4606 -1,2725 -1,0798 -,8933 -,7208	.0245 .0385 .0460 .0483 .0466	-1.9204 -1.6331 -1.3566 -1.1013 8739	3 3 3 3 3 3
8.5 8.6 8.7 2.8 8.9	5.5849 5.7004 58114 59198 6.0245	.1398 .1383 .1371 .1363 .1356	5.8859 5.3019 5.3146 5.3836 5.3897	11805 11305 10985 10642 10437	0173 0134 0100 0073 0058	2106 1512 1066 0738 0501	5676 4364 3277 3406 1725	.0428 .0365 .0301 .0841 .0184	6782 5149 3825 3783 1982	3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3
3.0 3.1 3.8 3.3 3.4	6.1281 6.2305 6.3320 6.4330 6.5336	.1352 .1349 .1347 .1346 .1345	5,3338 5,3366 5,3383 5,3395 5,3408	1.0292 1,0191 1.0122 1.0076 1.0047	0036 0034 0016 0010 0006	.0334 .0219 .0141 .0089 .0055	1211 0830 0560 0363 0833	.0139 .0098 .0073 .0043 .0089	1385 0946 0638 0413 0268	33355
3.5 3.6 3.7 3.8 3.9	6.6340 6.7348 68343 6.9344 7.0345	.1344 .1344 .1344 .1344 .1344	5.3406 5.3409 5.3410 5.3411 5.3412	1.0028 1.0017 1.0010 1.0005. 1.0003	0004 0002 0001 0001 .0000	0034 .0020 .0013 .0007 .0004	0147 0091 0055 0033 0019	.0020 .0012 .0007 .0004 .0003	0170 0106 0064 0040 0023	3 3 3 3 3 3 3
4-0 4-1 4-8 4-3 4-4	71345 72345 73345 74345 75345	.1344 .1344 .1344 .1344 .1344 .1344	5.3418 5.3418 5.3418 5.3418 5.3418 5.3418	1.0001 1.0001 1.0000 1.0000 1.0000	00000. 00000. 00000. 00000.	0003 0001 0000 0000 0000	0011 0005 0003 ·0001 0001	.0001 .0001 .0001 .0000 .0000	0014 0007 0005 0003 0003	5 5 5 5 5 5 5
4.5	7-6345	.1344	5.3412	1.0000	0000.	0000	.0000	.0000	0000	3

NACA IN 3028

			[Pr, 0.72;	γ, 1.40]		NACA	
η	Ih11	Ih ₁₂	^h 11	^h 12	hil	hi2	N
0 1934	0.0000 .0159 .0353 .0614 .0968	0.0000 .0009 .0020 .0034 .0053	0.1524 .1710 .2232 .3034 .4059	0.0085 .0097 .0126 .0163 .0200	0.0000 .3631 .6712 .9231 1.1174	0.0000 .0221 .0348 .0386 .0344	1 1111
.5 .6 .7 .8 .9	1432 2021 2742 3598 4583	.0074 .0098 .0123 .0146 .0167	.5249 .6544 .7886 .9215 1.0479	.0230 .0246 .0244 .0223 .0183	1,2526 1,3282 1,3448 1,3051 1,2139	.0236 .0078 0111 0309 0494	1 1 1 1
1.0 1.1 1.2 1.3 1.4	.5690 .6904 .8208 .9584 1.1012	.0182 .0191 .0193 .0187 .0172	1.1628 1.2624 1.3437 1.4051 1.4460	.0126 .0055 0024 0104 0182	1,0784 .9083 .7151 .5118 .3091	0648 0755 0806 0799 0737	1 1 1 1
1.5 1.6 1.7 1.8 1.9	1.3470 1.3940 1.5406 1.6854 1.8273	.0151 .0123 .0090 .0054 .0016	1.4673 1.4708 1.4589 1.4347 1.4015	0250 0306 0347 0372 0382	.1200 0470 1856 2925 3671	0629 0489 0331 0171 0021	1 1 1 1
20 21 22 23 24	1.9655 20996 2.2295 2.3551 2.4767	0022 0059 0094 0126 0155	1.3623 1.3201 1.2772 1.2356 1.1967	0377 0361 0335 0304 0269	4114 4291 4251 4044 3720	.0109 .0214 .0290 .0338 .0361	1 1 1 1 1
2.5 2.6 2.7 2.8 2.9	2-5946 2.7091 2.8208 2.9299 3.0371	0180 0201 0219 0234 0234	1.1614 1.1303 1.1035 1.0809 1.0622	0232 0197 0163 0133 0106	3326 2898 2467 2056 1679	.0363 .0348 .0321 .0286 .0248	1 1 1 1
3.0 3.1 3.2 3.3 3.4	3.1 4 2 5 3.2 4 6 6 3.3 4 9 6 3.4 5 1 9 3.5 5 3 4	0255 0263 0268 0272 0272 0276	1.0472 1.0352 1.0259 1.0187 1.0134	0083 0064 0049 0036 0026	1345 1057 0816 0619 0464	.0210 .0173 .0139 .0110 .0085	1 1 1 1
3.5 3.6 3.7 3.8 3.9	3.6546 3.7554 3.8559 3.9563 4.0565	0278 0279 0281 0281 0282	1.0094 1.0065 1.0044 1.0030 1.0020	0019 0013 0009 0006 0004	0338 0244 0173 0120 0082	.0064 .0048 .0035 .0025 .0018	1 1 1 1
4.0 4.1 4.2 4.3 4.4	4.1 567 4.2 568 4.3 568 4.4 569 4.5569	0282 0282 0283 0283 0283	1.00131.00081.00051.00031.0002	0003 0002 0001 0001 .0000	0056 0037 0024 0016 0010	.0012 .0008 .0006 .0004 .0002	1 1 1 1
4.5 4.6 4.7 4.8 4.9	4.6569 4.7569 4.8570 4.9570 5.0570	0283 0283 0283 0283 0283 0283	1.00011.00011.00011.00001.0000	.0000 .0000 .0000 .0000 .0000	0006 0004 0002 0001 .0000	.0002 .0001 .0001 .0000 .0000	1 1 1 1

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TABLE IV	SOLUTIONS OF	FIRST-ORDER	ENERGY	EQUATION	FOR	ZERO	HEAT	TRANSFER
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	TABLE IV CONCLINED. SOLUTIONS OF FIRST-ONDER ENERGY EQUATION FOR ZERO HEAT TRANSFER						
	·····	· · · · · · · · · · · · · · · · · · ·	[PF, 0	. /2; γ, 1.40]			مستعرف
	1n21	1h22	h ₂₁	^h 22	h21	^b 22	N
0 -1 M J 4 -4	0.0000 0162 0379 0695 1143	0.0000 .0013 .0031 .0056 .0089	0.1 526 1812 2597 .3774 .5239	0.0123 .0148 .0210 .0291 .0375	0.0000 .5533 .9986 1.3381 1.5753	0.0000 .0465 .0744 .0852 .0808	พพพพ พ
.5 .6 .7 8 .9	1749 2525 3477 .4600 .5885	.0130 .0178 .0230 .0282 .0331	.6892 .8638 1.0390 1.2067 1.3600	.0449 .0500 .0522 .0512 .0468	1,7149 1,7628 1,7267 1,6159 1,4419	.0639 .0379 .0061 0275 0595	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
1.0 1.1 1.2 1.3 1.4	.7314 .8864 10510 12224 1.3979	.0374 .0409 .0433 .0445 .0446	1.49341.60251.68471.73901.7660	.0394 .0297 .0183 .0063 0056	1.2181 9597 .6830 .4044 .1392	0867 1069 1185 1212 1153	8 8 8 8 8 8
1.5 1.6 1.7 1.8 1.9	1.5747 1.7507 1.9236 2.0920 2.2547	.0434 .0413 .0383 .0347 .0307	$\begin{array}{c} 1.7 & 6 & 7 & 7 \\ 1.7 & 4 & 7 & 4 \\ 1.7 & 0 & 8 & 9 \\ 1.6 & 5 & 6 & 8 \\ 1.5 & 9 & 5 & 3 \end{array}$	0165 0259 0332 0384 0414	0993 3013 4607 5753 6464	1023 0840 0627 0405 0191	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
2.0 2.1 2.2 2.3 2.4	2.4109 2.5604 2.7031 2.8394 2.9696	.0265 .0223 .0183 .0145 .0111	1.5288 1.4608 1.3943 1.3317 1.2743	0423 0415 0393 0362 0324	6782 6767 6490 6022 5432	0002 .0154 .0273 .0354 .0400	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
2,5 2,6 2,7 2,8 2,9	3.0 9 4 4 3.2 1 4 5 3.3 3 0 4 3.4 4 2 9 3.5 5 2 5	.0080 .0054 .0032 .0014 0001	1.2232 1.1788 1.1410 1.1095 1.0838	0283 0241 0202 0165 0133	4778 4107 3456 2851 2307	.0416 .0408 .0383 .0347 .0304	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
3.0 3.1 3.1 3.3 3.4 3.4	3.6598 3.7653 3.8693 3.9722 4.0743	0013 0022 0029 0035 0039	1.0631 1.0469 1.0343 1.0247 1.0176	0104 0081 0061 0046 0034	-1833 -1431 -1097 -0828 -0614	.0259 .0315 .0174 .0138 .0107	2 2 2 2 2 2 2 2
35 36 3.7 3.8 3.9	4.1758 4.2768 4.3776 4.4780 4.5783	0042 0044 0045 0046 0047	1.0123 1.0085 1.0058 1.0039 1.0026	0024 0017 0012 0008 0005	0448 0321 0227 0157 0107	.0081 .0061 .0044 .0032 .0022	2 2 2 2 2 2 2 3
4.0 41 42 43 44	4.6786 4.7787 4.8788 4.9788 5.0789	0047 0048 0048 0048 0048	1.00171.00111.00071.00041.0003	0004 0002 0001 0001 .0000	0072 0048 0031 0020 0014	.0016 .0011 .0007 .0005 .0003	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
4.5 4.6 4.7 4.8 4.9	5,1789 5,2789 5,3789 5,4789 5,5789	0048 0048 0048 0048 0048	1.00021.00011.00011.00011.00011.0000	-0000 -0000 -0000 -0000 -0000	0008 0005 0003 0008 0001	.0002 .0002 .0001 .0001 .0001	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2

ABLE IV. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ZERO HEAT TRANSFER

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	TADLE IV	concluded. SU	Pr, 0.7	-ORDER ENERGY E. 2; γ, 1.40]	UNTION FOR 255	NA	CAnn
η	Ih31	Ih ₃₂	h31	h32	h ₃₁	h ₃₂	N
0 मक्षज 4	0.0000 0165 0400 0761 1286	0.0000 .0016 .0038 .0072 .0118	0.1528 .1897 .2897 .4371 .6172	0.0146 .0183 .0275 .0396 .0524	0.00000 .7112 1.2623 1.6612 1.9186	0.0000 .0693 .1111 .1279 .1231	33333
.5 .6 .7 .8 .9	.2001 .2921 .4046 .5368 .6870	.0176 .0244 .0319 .0396 .0471	.8165 1.0229 1.2255 1.4154 1.5848	.0637 .0721 .0767 .0769 .0727	2.0473 2.0616 1.9764 1.8073 1.5710	.1009 .0663 .0242 0202 0624	3 3 3 3 3 3
1.0 1.1 1.2 1.3 1.4	.8528 1.0315 1.2199 1.4146 1.6125	.0540 .0599 .0646 .0678 .0697	17279 1.8407 1.9209 1.9683 1.9841	.0646 .0534 .0398 .0251 .0104	1.2849 0.9674 0.6371 0.3121 0.0090	0987 1261 1439 1486 1440	3 3 3 3 3 3
1.5 1.6 1.7 1.8 1.9	1.8105 2.0059 2.1966 2.3809 2.5574	.0700 .0691 .0670 .0641 .0606	1.9713 1.9339 1.8770 1.8056 1.7251	0034 0154 0253 0327 0375	2584 4801 6505 7680 8352	1304 1104 0863 0608 0359	33333
2.0 21 22 2.3 2.4	2,7257 2,8855 3,0368 3,1801 3,3162	.0567 .0526 .0487 .0449 .0414	1.6401 1.5548 1.4725 1.3958 1.3261	0399 0403 0390 0364 0330	8575 8425 7986 7343 6575	0135 .0054 .0202 .0307 .0372	3333
2.5 2.6 2.7 2.8 2.9	3.4456 3.5693 3.6881 3.8028 3.9141	.0383 .0356 .0333 .0314 .0298	1.2645 1.2112 1.1661 1.1286 1.0982	0291 0250 0310 0173 0140	5747 4914 4116 3381 2726	•0403 •0405 •0387 •0355 •0314	3 3 3 3 3 3
3.0 3.1 3.2 3.3 3.4	4.0 2 2 7 4.1 2 9 0 4.2 3 3 7 4.3 3 7 1 4.4 3 9 6	.0286 .0276 .0268 .0263 .0259	10738 10547 10399 10287 10204	0111 0086 0065 0049 0036	2159 1680 1285 0967 0715	.0269 .0225 .0183 .0146 .0114	3 3 3 3 3 3 3 3
3.5 3.6 3.7 3.8 3.9	4.5413 4.6425 4.7433 4.8438 4.9442	.0255 .0253 .0252 .0251 .0251 .0250	1.0142 1.0098 1.0066 1.0044 1.0029	0026 0018 0013 0009 0006	0520 0372 0263 0181 0124	.0087 .0065 .0048 .0034 .0024	3 3 3 3 3 3
4.0 4.1 4.2 4.3 4.4	5.0 4 4 4 5.1 4 4 6 5.2 4 4 7 5.3 4 4 8 5.4 4 4 8	.0249 .0249 .0249 .0249 .0249 .0249	1.00191.00121.00081.00051.0003	0004 0002 0001 0001 .0000	0083 0055 0036 0023 0014	.0017 .0012 .0008 .0005 .0004	3 3 3 3 3 3 3
45 46 47 48 49	5.5448 5.6448 5.7448 5.8449 5.9449	.0249 .0249 .0249 .0249 .0249 .0249	1.00021.00011.00011.00011.00011.0000	.0000 .0000 .0000 .0000 .0000	0009 0005 0003 0002 0001	.0003 .0002 .0001 .0000 .0000	3 3 3 3 3

TABLE IV. - Concluded. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR ZERO HEAT TRANSFER

ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]	ฎ
(a) The function $IH(\eta) =$	∫`H aη

		(a) The	function IH(η)	≖ ∫ H dη 0	NA	CA.
η	щл	IH12	^{III} 13	\mathbb{H}_{14}	IH ₁₅	N
0.1.2.3.4	0.0000 .0014 .0078 .0223 .0473	0.0000 .0001 .0005 .0013 .0025	0.0.000 0.013 0027 0010 .0059	0.0000 .0278 .1111 .2494 .4416	00000 0267 1058 2396 4238	1 111 1
.5 .6 .7 .8 .9	.0848 .1359 .2014 .2813 .3751	.0042 .0061 .0082 .0103 .0120	.0192 .0394 .0660 .0982 .1347	6859 9792 13173 16949 21057	6571 9357 12549 16087 19901	1 1 1 1
1.0 1.1 1.2 1.3 1.4	.4818 .5999 .7277 .8631 1.0041	.0134 .0141 .0141 .0134 .0138	.1739 .2141 .2535 .2907 .3244	25422 29965 34601 39245 43816	2.3 9 1 4 2.8 0 4 3 3.2 2 0 5 3.6 3 2 0 4.0 3 1 2	1 1 1 1
1.5 1.6 1.7 1.8 1.9	1.1486 1.2946 1.4167 1.5845 1.7258	.0096 .0067 .0034 0002 0041	.3537 .3780 .3970 .4111 .4206	48235 52437 56366 59980 63250	4.4117 4.7679 5.0958 5.3924 5.6563	1 1 1 1
2.0 21 22 23 2.4	1.8 6 3 7 1.9 9 7 6 2.1 2 7 2 2.2 5 2 7 2.3 7 4 2	0079 0116 0151 0183 0212	.4260 .4281 .4277 .4255 .4221	6.6161 6.8712 70912 72780 7.4341	5.8874 6.0863 6.2550 6.3956 6.5111	1 1 1 1
25 26 27 28 29	2.4920 2.6065 2.7181 2.8273 2.9344	0237 0258 0276 0291 0303	.4181 .4138 .4097 .4059 .4025	7.5625 7.6666 7.7497 7.8149 7.8654	6.6044 6.6787 6.7370 6.7820 6.8162	1 1 1 1
3.0 31 32 33 3.4	3.0 398 3.1 4 39 3.2 4 6 9 3.3 4 9 2 3.4 5 0 8	0313 0320 0325 0330 0333	3996 3972 3953 3938 3926	7.9040 7.9329 7.9543 7.9699 7.9811	6.8 4 1 8 6.8 6 0 7 6.8 7 4 5 6.8 8 4 4 6.8 9 1 3	1 1 1 1
3.5 3.6 3.7 3.8 3.9	3.5519 3.6527 3.7532 3.8536 3.9538	0335 0337 0338 0339 0339	.3917 .3911 .3906 .3903 .3901	7.9890 7.9946 7.9984 8.0009 8.0026	68962 68995 69017 69032 69042	1 1 1 1
4.0 4.1 4.2 4.3 4.4	4.0540 4.1541 4.2542 4.3542 4.4542	0339 0340 0340 0340 0340 0340	.3899 .3898 .3898 .3898 .3897 .3897	8.0 0 3 8 8.0 0 4 5 8.0 0 5 0 8.0 0 5 3 8.0 0 5 4	6.9049 69053 69055 69057 69057 69058	1 1 1 1
4.5 4.6 4.7 4.8 4.9	4.5 5 4 3 4.6 5 4 3 4.7 5 4 3 4.8 5 4 3 4.9 5 4 3	0340 0340 0340 0340 0340 0340	.3897 .3897 .3897 .3897 .3897 .3897	8.0 0 5 6 8.0 0 5 6 8.0 0 5 7 8.0 0 5 7 8.0 0 5 7 8.0 0 5 7	6.9058 69059 69059 69059 69059 69059	1 1 1 1
5.0	5.0543	0340	.3897	8.0057	69059	1

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ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

	•	L	(] (31.0 (11	*• J		
	(a) Continued.	The function	$IH(\eta) = \int_0^{\eta} H d\eta$	NA	A
ŋ	IH21	IH ₂₂	^{III} 23	IH ₂₄	IH ₂₅	N
0 .1 .2 .3 .4	$\begin{array}{c} 0.0000\\ .0019\\ .0109\\ .0316\\ .0672 \end{array}$	0.0000 .0002 .0009 .0025 .0051	0.0000 .0002 .0055 .0210 .0499	0.0000 .0435 .1738 .3899 .6899	0.0000 .0509 .2033 .4559 .8055	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
5.67.89	.1199 .1910 .2808 .3888 .5137	.0086 .0128 .0176 .0224 .0270	.0936 .1520 .2237 .3063 .3969	1.0698 1.5239 2.0449 2.6232 3.2481	1.2465 1.7710 2.3685 3.0266 3.7311	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
1.0 1.1 1.2 1.3 1.4	.6538 .8067 .9696 1.1397 1.3142	.0312 .0345 .0367 .0379 .0378	.4922 .5887 .6832 .7729 .8554	3.9 0 7 3 4.5 8 8 1 5.2 7 7 4 5.9 6 2 3 6.6 3 0 8	4.4668 5.2180 59694 6.7064 7.4161	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
1.5 1.6 1.7 1.8 1.9	1.4903 1.6657 1.8383 2.0063 2.1688	.0366 .0344 .0314 .0278 .0238	.9290 .9927 1.0460 1.0892 1.1229	7.2721 7.8771 8.4384 8.9508 9.4112	8.0874 8.7115 9.2819 9.7947 10.2484	N N N N N
2.0 2.1 2.2 2.3 2.4	2.3249 2.4743 2.6169 2.7531 2.8833	.0196 .0154 .0113 .0075 .0041	1.1 481 1.1 661 1.1 780 1.1 852 1.1 889	9.8183 10.1727 10.4764 10.7328 10.9458	1 0.6 4 3 3 1 0.9 8 1 6 1 1.2 6 6 9 1 1.5 0 3 9 1 1.6 9 7 7	N N N N N
2.5 2.6 2.7 2.8 2.9	3.0081 3.1282 3.2441 3.3566 3.4662	.0010 0016 0038 0056 0071	1,1 900 1,1 894 1,1 878 1,1 857 1,1 834	11.120211.260711.372411.459711.5270	1 1,8 5 3 8 1 1,9 7 7 6 1 2.0 7 4 4 1 2.1 4 8 9 1 2.2 0 5 4	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
3.0 3.1 3.2 3.3 3.4	3.5735 3.6790 3.7830 3.8859 3.9880	0083 0092 0099 0105 0109	1.1812 11792 11775 11761 11750	11.578111.616411.644611.665011.6797	1 2.2 4 7 7 1 2.2 7 8 8 1 2.3 0 1 4 1 2.3 1 7 6 1 2.3 2 9 0	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
3.5 3.6 3.7 3.8 3.9	4.0895 4.1905 4.2912 4.3917 4.4920	0111 0114 0115 0116 0117	11741 11734 11729 11726 11723	11.690011.697211.702111.705411.7076	123369123423123460123484123500	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
4.0 4.1 4.2 4.3 4.4	4.5922 4.6924 4.7924 4.8925 4.9925	0117 0118 0118 0118 0118 0118	1.1722 1.1720 1.1720 1.1719 1.1719	11.709111.710011.710611.711011.711011.7112	12.3510123517123521123523123523123525	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
4.5 4.6 4.7 4.8 4.9	5.0926 5.1926 5.2926 5.3926 5.4926	0118 0118 0118 0118 0118	1.1719 1.1719 1.1719 1.1719 1.1719 1.1719	$\begin{array}{r} 11.7114\\ 11.7115\\ 11.7115\\ 11.7115\\ 11.7115\\ 11.7116\end{array}$	123526123526123527123527123527123527123527	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
5-0 5.1 5.2	5.5926 5.6926 5.7926	0118 0118 0118	1,1719 1,1719 1,1719	11.7116 11.7116 11.7116	1 2 3 5 2 7 1 2 3 5 2 7 1 2 3 5 2 7 1 2 3 5 2 7	2 2 2 2

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ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; r, 1.40]

(a) Concluded. The function $IH(\eta) = \int_{H}^{\eta} H d\eta$

		· · · · · · · ·		J_0^{-1}	NACA	•
η	^{IE} 31	^{IH} 32	IH33	IH ₃₄	^{IE} 35	N
0 .1 .2 .3 .4	0.0000 .0023 .0135 .0391 .0830	0.0000 .0002 .0013 .0037 .0074	0.0000 .0023 .0153 .0458 .0977	0,0000 0569 2274 .5101 .9015	0.0000 .0727 .2905 .6510 1.1488	3 3 3 3 3 3 3
.5 .6 .7 .8 .9	.1475 .2338 .3417 .4703 .6178	.0126 .0189 .0259 .0333 .0406	1720 2680 3830 5132 6541	1.3958 1.9848 2.6573 3.4002 4.1984	1.7749 2.5165 3.3574 4.2786 5.2591	33333
1.0 1.1 1.2 1.3 1.4	.7815 .9586 11458 13397 15369	.0473 .0530 .0576 .0607 .0625	.8010 .9490 1.0936 1.2309 1.3577	5.0358 5.8954 6.7605 7.6150 8.4443	6.2769 7.3100 8.3370 9.3387 10.2979	33333
1.5 1.6 1.7 1.8 1.9	1.7345 1.9296 2.1201 2.3041 2.4806	.0628 .0618 .0597 .0568 .0532	14719 15720 16575 17285 17860	9.2354 9.9778 10.6630 11.2857 11.8426	1 1.2006 1 2.0357 1 2.7956 1 3.4761 1 4.0758	3 5 5 5 5 5
2.0 2.1 2.2 2.3 2.4	2.6488 2.8085 2.9598 3.1032 3.2392	.0493 .0453 .0413 .0375 .0341	1.8 31 1 1.8 65 5 1.8 9 0 7 1.9 0 8 6 1.9 2 0 7	1 2.3 3 3 0 1 2.7 5 8 2 1 3.1 2 1 4 1 3.4 2 6 8 1 3.6 7 9 8	14.5961 15.0405 15.4144 15.7242 15.9770	33333
2.5 2.6 2.7 2.8 2.9	3.3686 3.4923 3.6111 3.7258 3.8371	.0310 .0282 .0259 .0240 .0225	1.9284 1.9329 1.9352 1.9360 1.9359	1 3.8 8 6 3 1 4.0 5 2 3 1 4.1 8 3 7 1 4.2 8 6 3 1 4.3 6 5 2	16.1803 163412 16.4669 16.5635 16.6367	3 3 3 3 3 3 3 3
3.0 3.1 3.2 3.3 3.4	3.9456 4.0520 4.1567 4.2601 4.3626	.0212 .0202 .0195 .0189 .0185	1.9353 1.9344 1.9336 1.9328 1.9321	1 4.4 2 5 0 1 4.4 6 9 7 1 4.5 0 2 5 1 4.5 2 6 3 1 4.5 4 3 3	16.6914 16.7316 16.7608 16.7816 16.7963	3 3 3 3 3 3 3 3
3.5 3.6 3.7 3.8 3.9	4.4 6 4 3 4.5 6 5 5 4.6 6 6 3 4.7 6 6 8 4.8 6 7 2	.0182 .0180 .0178 .0177 .0176	1.9 31 5 1.9 31 1 1.9 30 8 1.9 30 5 1.9 30 4	1 4.5 5 5 3 1 4.5 6 3 6 1 4.5 6 9 3 1 4.5 7 3 1 1 4.5 7 5 6	16.8066 16.8135 16.8182 16.8214 16.8234	33333
4.0 4.1 4.2 4.3 4.4	4.9674 50676 51677 52677 53678	.0175 .0175 .0175 .0175 .0175	1.9 30 3 1.9 302 1.9 301 1.9 301 1.9 301 1.9,301	1 4.5 7 7 3 1 4.5 7 8 4 1 4.5 7 9 0 1 4.5 7 9 5 1 4.5 7 9 7	168247 168256 168261 168264 168264 168266	33339
4.5 4.6 4.7 4.8 4.9	5.4678 5.5678 5.6678 5.7678 5.8678	.0174 .0174 .0174 .0174 .0174	1.9301 1.9301 1.9301 1.9301 1.9301 1.9301	1 4.5 7 9 9 1 4.5 8 0 0 1 4.5 8 0 1 1 4.5 8 0 1 1 4.5 8 0 1 1 4.5 8 0 1	1 6.8 2 6 8 1 6.8 2 6 8 1 6.8 2 6 9 1 6.8 2 6 9 1 6.8 2 6 9 1 6.8 2 6 9	NNNN N
5.0 5.1 5.2 5.3	5.9678 6.0678 6.1678 6.2678	.0174 .0174 .0174 .0174	1.9301 1.9301 1.9301 1.9301	14.5801 14.5801 14.5801 14.5801	16.8269 16.8269 16.8269 16.8269	3333

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TABLE V. - Continued. SOLUTIONS OF FIRST-ORDER ENERGY EQUATION FOR

ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

(b) The function $H(\eta)$

η	H ₁₁	H ₁₂	H ₁₃	H ₁₄	H ₁₅	N
0	0.0000	0.0000	0.0000	0.0000	0.0000	11111
1	.0333	.0020	0198	.5556	.5343	
2	.1000	.0057	0028	1.1091	1.0659	
3	.1942	.0102	.0404	1.6554	1.5884	
.4	.3100	.0147	.1000	2.1869	2.0923	
.5	.4414	.0183	.1674	2.6938	2.5 6 6 4	11111
.6	.5825	.0206	.2350	3.1648	2.9 9 8 3	
.7	.7272	.0210	.2962	3.5885	3.3 7 6 0	
.8	.8696	.0194	.3462	3.9534	3.6 8 8 4	
.9	1.0044	.0159	.3813	4.2493	3.9 2 6 7	
1.0	1.1268	.0106	.3997	4.4679	4.0846	11111
1.1	1.2328	.0039	.4008	4.6038	4.1593	
1.2	1.3197	0037	.3856	4.6543	4.1516	
1.3	1.3857	0115	.3563	4.6205	4.0655	
1.4	1.4306	0190	.3160	4.5068	3.9088	
1.5	1.4552	0257	.2682	4.3211	3.6 9 1 6	11111
1.6	1.4613	0312	.2168	4.0738	3.4 2 6 2	
1.7	1.4516	0351	.1653	3.7773	3.1 2 5 9	
1.8	1.4291	0376	.1167	3.4454	2.8 0 4 3	
1.9	1.3973	0384	.0733	3.0920	2.4 7 4 3	
2.0	13592	0379	.0367	2.7305	2.1478	11111
2.1	13178	0362	.0075	2.3731	1.8344	
2.2	12755	0336	0142	2.0301	1.5419	
2.3	12344	0305	0291	1.7096	1.2756	
2.4	11958	0269	0380	1.4174	1.0390	
2.5	1.1608	0233	0420	11570	.8332	11111
2.6	1.1299	0197	0423	9301	.6580	
2.7	1.1032	0163	0401	.7363	.5117	
2.8	1.0807	0133	0362	.5740	.3920	
2.9	1.0621	0106	0314	.4408	2958	
3.0	1.0471	0083	0264	3335	.2199	11111
3.1	1.0352	0064	0216	2485	.1611	
3.2	1.0259	0049	0172	1824	.1163	
3.3	1.0188	0036	0134	1320	.0827	
3.4	1.0134	0026	0101	0940	.0579	
3.5 3.6 3.7 3.8 3.9	$\begin{array}{c} 1.0 & 0 & 9 & 4 \\ 1.0 & 0 & 6 & 6 \\ 1.0 & 0 & 4 & 5 \\ 1.0 & 0 & 3 & 0 \\ 1.0 & 0 & 2 & 0 \end{array}$	0019 0013 0009 0006 0004	0075 0055 0039 0027 0018	.0660 .0457 .0311 .0209 .0138	.0400 .0272 .0183 .0121 .0079	11111
4.0	1.0013	0003	0012	.0090	.0051	11111
4.1	1.0008	0002	0007	.0058	.0032	
4.2	1.0005	0001	0004	.0037	.0020	
4.3	1.0003	0001	0002	.0023	.0012	
4.4	1.0002	.0000	0001	.0014	.0007	
4.5 4.6 4.7 4.8 4.9	$\begin{array}{c} 1.0 & 0 & 0 \\ 1.0 & 0 & 0 \\ 1.0 & 0 & 0 \\ 1.0 & 0 & 0 \\ 1.0 & 0 & 0 \\ 1.0 & 0 & 0 \end{array}$.0000 .0000 .0000 .0000	.0000 .0000 .0000 .0000 .0000	.0009 .0005 .0003 .0002 .0001	.0004 .0003 .0001 .0001 .0000	コーコー
5.0	1.0000	.0000	.0000	.0000	.0000	1

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ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; r, 1.40]

		(b) Contin	ued. The funct	ion H(η)	NAC	And
ŋ	H ₂₁	H ₂₂	H ₂₃	H24	H ₂₅	N
0 .1 .2 .3 .4	0.0000 .0465 .1424 .2766 .4384	0.0000 .0039 .0115 .0210 .0306	0.0000 .0170 .0969 .2183 .3622	0.0000 .8695 1.7347 25854 3.4069	0.0000 1.0177 2.0287 3.0179 3.9641	N N N N N
.5 .6 .7 .8 .9	.6176 .8046 .9905 1.1676 1.3288	.0391 .0452 .0483 .0480 .0480	•5118 .6534 .7760 .8716 .9352	4.1 8 1 3 4.8 8 9 8 5.5 1 3 4 6.0 3 5 3 6.4 4 1 5	4.8429 5.6294 6.3007 6.8378 7.2269	8 N N N N
1.0 1.1 1.2 1.3 1.4	1.4688 1.5833 1.6699 1.7278 1.7575	.0375 .0281 .0171 .0054 0063	.9646 .9605 .9255 .8642 .7827	6.7222 6.8722 6.8917 6.7862 6.5658	7.4606 7.5379 7.4650 7.2538 6.9215	N N N N N
1.5 1.6 1.7 1.8 1.9	1.76141.74271.70551.65431.5936	0170 0263 0335 0386 0415	.6874 .5851 .4819 .3831 .2928	6.2451 5.8418 5.3758 4.8677 4.3379	6.4894 5.9810 5.4210 4.8333 4.2402	N N N N N N N N
2.0 2.1 2.2 2.3 2.4	1.5276 1.4533 1.3937 1.3312 1.2740	0424 0416 0394 0362 0324	.2137 .1472 .0937 .0524 .0222	3.8052 3.2861 2.7941 2.3394 1.9290	3.6610 3.1114 2.6033 2.1448 1.7402	8 N N N N
2.5 2.6 2.7 2.8 2.9	12230 11787 11409 11095 10838	0283 0242 0202 0165 0133	.0014 0118 0192 0224 0226	1.5665 1.2531 .9874 .7664 .5861	1.3907 1.0947 .8489 .6486 .4882	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
3.0 3.1 3.2 3.3 3.4	1.0 6 3 1 1.0 4 6 9 1.0 3 4 3 1.0 2 4 7 1.0 1 7 6	0105 0081 0062 0046 0034	0211 0185 0156 0127 0100	.4416 .3278 .2398 .1729 .1228	.3621 .2647 .1907 .1354 .0947	8 8 8 8 8 8
3.5 3.6 3.7 3.8 3.9	1.0123 1.0085 1.0058 1.0039 1.0026	0025 0017 0012 0009 0006	0076 0057 0042 0030 0021	.0859 .0592 .0402 .0269 .0178	.0653 .0444 .0297 .0196 .0128	N N N N N
4.0 4.1 4.2 4.3 4.4	$\begin{array}{c} 1.0 \ 0 \ 1 \ 7 \\ 1.0 \ 0 \ 1 \ 1 \\ 1.0 \ 0 \ 0 \ 7 \\ 1.0 \ 0 \ 0 \ 4 \\ 1.0 \ 0 \ 0 \ 3 \end{array}$	0004 0003 0002 0001 0001	$\begin{array}{r}0014 \\0010 \\0006 \\0004 \\0003 \end{array}$.0116 .0074 .0047 .0029 .0018	.0082 .0052 .0032 .0020 .0012	N N N N N
4.5 4.6 4.7 4.8 4.9	$\begin{array}{c} 1.0\ 0\ 0\ 2\\ 1.0\ 0\ 0\ 1\\ 1.0\ 0\ 0\ 1\\ 1.0\ 0\ 0\ 1\\ 1.0\ 0\ 0\ 0\end{array}$	$\begin{array}{c}0001\\0001\\ .0000\\ .0000\\ .0000\\ .0000\end{array}$	0002 0002 0001 0001 0001	.0011 .0007 .0004 .0002 .0001	.0007 .0004 .0002 .0001 .0001	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
5.0 5.1 5.2 5.3	1.0000 1.0000 1.0000		$ \begin{array}{c} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 \\ \end{array} $.0001 .0001 .0001	.0001 .0001 .0001	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8

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ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

		(b) Conclud	led. The functi	on H(ŋ)	NA NA	CA
η	H ₃₁	H ₃₂	H ₃₃	H ₃₄	^H 35	' N
0 .1 .2 .3 .4	0.0000 .0473 .1767 .3423 .5390	0.0000 .0056 .0167 .0306 .0449	0.0000 .0624 .2090 .4079 .6310	0.0000 1.1383 2.2695 3.3778 4.4407	0.0 0 0 0 1.4 5 4 5 2.8 9 7 3 4.3 0 3 3 5.6 3 7 8	3 3 3 3 3 3 3
.5 .6 .7 .8 .9	.7529 .9719 11853 13840 1.5607	.0576 .0673 .0729 .0739 .0705	.8547 1.0601 1.2330 1.3638 1.4473	5.4327 6.3275 7.1008 7.7320 8.2059	6.8629 7.9423 8.8443 9.5447 10.0285	3 3 3 3 3 3
1.0 1.1 1.2 1.3 1.4	1.7097 1.8271 1.9109 1.9609 1.9788	.0629 .0520 .0388 .0244 .0099	1.4819 1.4697 1.4153 1.3255 1.2084	8.5131 8.6511 8.6240 8.4422 8.1217	102903 103346 101744 98307 93302	3 3 3 3 3 3 3
1.5 1.6 1.7 1.8 1.9	1.9675 1.9313 1.8751 1.8043 1.7242	0037 0157 0255 0328 0376	1.0730 .9280 .7816 .6406 .5104	7.6829 7.1496 6.5470 5.9009 5.2359	8.7040 7.9850 7.2067 6.4007 5.5957	3 3 3 3 3
2.0 2.1 2.2 2.3 2.4	1.6395 1.5544 1.4723 1.3956 1.3260	0400 0403 0390 0364 0330	.3947 .2954 .2132 .1475 .0968	4.5744 3.9355 3.3344 2.7827 2.2874	4.8162 4.0818 3.4068 2.8007 2.2679	3 3 3 3 3 3 3
2.5 2.6 2.7 2.8 2.9	1.2644 1,2111 1,1660 1,1286 1.0982	0291 0250 0211 0173 0140	.0591 .0323 .0141 .0026 0041	1,8523 1,4776 1,1614 .8994 .6862	1.8092 1.4220 1.1012 .8403 .6319	3 3 3 3 3 3 3 3 3
3.0 3.1 3.2 3.3 3.4	1.0738 1.0547 1.0399 1.0287 1.0204	0111 0086 0066 0049 0036	0075 0086 0084 0075 0062	.5160 .3823 .2791 .2008 .1424	.4683 .3420 .2462 .1747 .1221	3 3 3 3 3 3 3
3.5 3.6 3.7 3.8 3.9	1.0142 1.0098 1.0066 1.0044 1.0029	0026 0019 0013 0009 0006	0050 0038 0028 0020 0014	.0994 .0684 .0464 .0310 .0204	.0841 .0571 .0382 .0252 .0164	3 3 3 3 3 3
4.0 4.1 4.2 4.3 4.4	$\begin{array}{c} 1.0\ 01\ 9\\ 1.0\ 01\ 2\\ 1.0\ 00\ 8\\ 1.0\ 00\ 5\\ 1.0\ 00\ 3\\ \end{array}$	0005 0003 0002 0002 0002	0009 0006 0004 0002 0001	.0133 .0085 .0054 .0033 .0021	.0105 .0066 .0041 .0025 .0015	3 3 3 3 3 3
4.5 4.6 4.7 4.8 4.9	$\begin{array}{c} 1.0002\\ 1.0001\\ 1.0001\\ 1.0001\\ 1.0001\\ 1.0000\end{array}$	0001 .0000 .0000 .0000 .0000	.0000 .0000 .0000 .0000 .0000	.0012 .0007 .0004 .0003 .0002	.0009 .0005 .0003 .0002 .0002	3 3 3 3 3 3
5.0 5.1 5.2 5.3	$\begin{array}{c} 1.0\ 0\ 0\ 0\\ 1.0\ 0\ 0\ 0\\ 1.0\ 0\ 0\ 0\\ 1.0\ 0\ 0\ 0\end{array}$.0000 .0000 .0000	.0000. 0000. 0000. 0000.	.0001 .0001 .0001 .0000	0000. 0000. 0000. 0000.	3 3 3 3 3

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ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; γ , 1.40]

	(c) The function $H'(\eta)$				NA	CA,
η 	H'11	<u> </u>	E'13	H ₁₄	H'15	N
0 •1 .2 .3 .4	0.1479 ·5096 .8138 1.0598 1.2464	0.0082 .0302 .0427 .0462 .0416	-0.4201 .0042 .3179 .5298 .6492	5.5574 5.5512 5.5101 5.4040 5.2093	5.3452 5.3370 5.2837 5.1497 4.9101	1 1 1 1 1 1 1
.5 .6 .7 .8	1.3728 1.4386 1.4450 1.3949 1.2932	.0303 .0139 0055 0259 0450	.6865 .6534 .5632 .4301 .2695	4.9087 4.4931 3.9610 3.3197 2.5847	4.5506 4.0675 3.4675 2.7665 1.9888	1 1 1 1
1.0 1.1 1.2 1.3 1.4	1.1477 .9682 .7662 .5543 .3450	0609 0722 0778 0775 0717	.0968 0731 2268 3538 4467	1.7794 .9329 .0789 7475 -1.5121	1.1652 .3306 4790 -12289 -1.8888	1 1 1 1
1.5 1.6 1.7 1.8 1.9	.1495 0230 1663 2771 3550	0613 0476 0321 0163 0014	5020 5203 5051 4630 4018	-2.1847 -2.7412 -3.1653 -3.4497 -3.5959	-2.4349 -2.8515 -3.1320 -3.2784 -3.3005	1 1 1 1
2.0 2.1 2.2 2.3 2.4	4020 4220 4196 4003 3690	.0115 .0218 .0293 .0341 .0363	3296 2542 1817 1169 0625	-3.6132 -3.5176 -3.3294 -3.0715 -2.7669	-32145 -30405 -28009 -25181 -22129	1 1 1 1
2.5 2.6 2.7 2.8 2.9	3303 2882 2456 2048 1673	.0364 .0348 .0321 .0287 .0248	0198 .0114 .0523 .0444 .0496	-2.4374 -2.1021 -1.7765 -1.4722 -1.1972	-1.9033 -1.6038 -1.3251 -1.0743 8550	1 1 1 1 1 1 1
3.0 3.1 3.2 3.3 3.4	-1340 -1054 -0814 -0617 -0458	.0210 .0173 .0139 .0110 .0085	.0497 .0465 .0412 .0353 .0291	9558 7495 5775 4373 3256	6684 5135 3878 2879 2102	1 1 1 1 1
3.5 3.6 3.7 3.8 3.9	0338 0241 0175 0122 0083	.0064 .0048 .0035 .0025 .0018	.0233 .0181 .0137 .0102 .0074	2384 1716 1217 0844 0580	1511 1067 0743 0506 0342	1 1 1 1 1
4.0 4.1 4.2 4.3 4.4	0056 0037 0024 0015 0010	.0012 .0008 .0006 .0004 .0002	.0053 .0037 .0025 .0017 .0012	0392 0261 0169 0110 0069	0227 0149 0095 0061 0038	1 1 1 1 1
4.5 4.6 4.7 4.8 4.9	0006 0004 0002 0001 .0001	.0002 .0001 .0001 .0000 .0000	.0008 .0005 .0003 .0001 .0001	0044 0026 0016 0009 0005	0023 0014 0009 0005 0003	1 1 1 1

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5.0 5.1 5.2 5.3

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,

ARBITRARY RATES OF HEAT TRANSFER

[Pr, 0.72; Y, 1.40]

		(d) Cont	inued. The functi	on H'(ŋ)	NA	CA
ŋ	H'21	H'22	H'23_	H'24	^н '25	N
0 .1 2 3 .3 .4	01802 .7308 11688 1.4974 1.7214	0.0145 .0608 .0882 .0981 .0926	-0.2200 .5214 1.0403 1.3564 1.4932	8.6985 8.6855 8.6008 8.3886 8.0108	10.1820 10.1618 10.0325 9.7161 9.2336	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
.5.6.7.8.9	1.8462 1.8787 1.8273 1.7019 1.5142	.0745 .0472 .0143 0205 0536	1.4771 1.3366 1.1015 .8017 .4665	7.4 4 6 4 6.6 9 0 5 5.7 5 3 9 4.6 6 0 7 3.4 4 7 0	8.3668 7.3248 6.0700 4.6490 3.1207	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
1.0 1.1 1.2 1.3 1.4	12780 10086 .7223 .4356 .1637	0819 1030 1154 1186 1133	.1237 -2021 -4901 -7244 -8953	2.1577 .8433 4432 -1.6506 _2.7328	1.5507 .0071 -1.4456 -2.7499 -3.8598	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
1.5 1.6 1.7 1.8 1.9	0803 2868 4498 5671 6404	1008 0829 0618 0398 0187	9989 -1.0374 -1.0179 9513 8506	-3.6519. -4.3809 -4.9051 -5.2224 -5.3422	-4.7432 -5,3830 -5.7775 -5.9390 -5.8915	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
2.0 2.1 2.2 2.3 2.4	6738 6735 6467 6007 5421	.0001 .0157 .0275 .0355 .0401	7292 5996 4722 3547 2522	-5.2842 -5.0754 -4.7476 -4.3343 -3.8683	-5.6674 -5.3047 -4.8427 -4.3198 -3.7703	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
2.5 2.6 2.7 2.8 2.9	4770 4102 3453 2848 2305	.0417 .0409 .0383 .0347 .0304	1673 1003 0503 0150 .0081	-3,3791 -2,8920 -2,4271 -1,9985 -1,6157	-3.2234 -2.7019 -2.2219 -1.7939 -1.4225	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2
3.0 3.1 3.2 3.3 3.4	1832 1430 1097 0827 0613	0258 0215 0174 0138 0107	.0216 .0282 .0298 .0284 .0284 .0253	-12829 -10009 7675 5787 4292	-1.1084 8490 6394 4737 3452	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
3.5 3.6 3.7 3.8 3.9	0448 0321 0227 0157 0107	.0081 .0060 .0044 .0031 .0022	.0214 .0174 .0136 .0103 .0076	3131 2245 1587 1096 0750	2476 1746 1214 0826 0556	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8
4.0 41 42 43 44	-0072 -0048 -0031 -0020 -0012	0015 0010 0007 0004 0002	.0056 .0039 .0027 .0018 .0012	0506 0336 0217 0140 0088	0369 0242 0154 0098 0061	8 8 8 8 8 8
45 46 4.7 4.8 49	-0008 -0005 -0003 -0002 -0001	0001 0001 0000 0000 0000 0000	.0008 .0005 .0003 .0001 .0000	-0055 -0034 -0021 -0012 -0007	0038 0023 0014 0008 0004	8 N N N N
5.0 5.1 5.2 5.3	0000. 0000. 0000. 0000.	0000 0000 0000 0000	0000. 0000. 0000. 0000.	0003 0002 0001 0000	0002 0001 0001 .0000	8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8

ARBITRARY RATES OF HEAT TRANSFER

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[Pr,	0.72;	r,	1.40]

		(c) Concluded. The function H'(η)				
η	H'31	^H 32	^E 33	Н <u>'</u> 34	^H 35	N
0 नुकुर्भ र ्	0.2042 .9113 1.4518 1.8353 2.0745	0.0195 .0884 .1292 .1446 .1380	0.0899 1.1008 1.7774 2.1532 2.2698	1 1.3 8 7 9 1 1.3 6 6 7 1 1.2 3 0 6 1 0.8 9 7 5 1 0.3 1 8 8	14.5535 14.5183 14.2961 13.7641 12.8617	どうちょう
.5.6.7.8.9	2.1837 2.1783 2.0743 1.8879 1.6362	.1139 .0774 .0336 0125 0562	21731 19111 15308 10768 5900	9.4767 8.3784 7.0529 5.5460 3.9150	1 1.5 8 0 7 9.9 5 3 8 8.0 4 4 8 5.9 3 8 0 3.7 2 9 0	3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3
1.0	1.3367	0937	.1064	2.2247	1.5156	33333
1.1	1.0079	1222	3431	.5423	6085	
1.2	.6682	1399	7340	-1.0666	-2.5599	
1.3	.3357	1463	-10489	-2.5421	-42699	
1.4	.0267	1422	-1.2773	-3.8336	-5.6878	
1.5	2454	1292	-1.4162	-4.9022	- 6.7821	33333
1.6	4707	1095	-1.4697	-5.7230	- 7.5412	
1.7	6437	0857	-1.4474	-6.2858	- 7.9725	
1.8	7632	0603	-1.3635	-6.5948	- 8.0995	
1.9	8318	0356	-1.2344	-6.6674	- 7.9594	
2.0	8552	01 32	-1.0770	-6.5313	-7.5980	33333
2.1	8409	.0056	9070	-6.2221	-7.0666	
2.2	7976	.0203	7377	-5.7794	-6.4171	
2.3	7336	.0307	5792	-5.2441	-5.6987	
2.4	6570	.0375	4383	-4.6551	-4.9552	
2.5	5743	.0403	3187	-4.0470	-4.2229	33339
2.6	4911	.0405	2214	-3.4489	-3.5301	
2.7	4115	.0387	1454	-2.8833	-2.8963	
2.8	3380	.0355	0887	-2.3660	-2.3337	
2.9	2726	.0314	0481	-1.9066	-1.8474	
3.0	2159	.0269	0208	-1.5094	-1.4374	ろうろろろ
3.1	1680	.0225	0034	-1.1745	-1.0996	
3.2	1285	.0183	.0065	8984	8272	
3.3	0967	.0146	.0114	6759	6123	
3.4	0716	.0114	.0130	5003	4460	
3.5	0422	.0086	.0124	3642	3196	ろうちろう
3.6	0373	.0064	.0109	2607	2253	
3.7	0263	.0047	.0090	1839	1566	
3.8	0182	.0033	.0070	1267	1064	
3.9	0124	.0023	.0053	0865	0715	
4.0	0083	.0016	.0039	0583	0476	33333
4.1	0055	.0011	.0028	0387	0311	
4.2	0035	.0007	.0019	0249	0198	
4.3	0023	.0004	.0013	0160	0125	
4.4	0014	.0002	.0009	0101	0078	
4.5	0009	.0002	.0006	0063	0048	ろうろうろ
4.6	0005	.0002	.0004	0038	0029	
4.7	0003	.0001	.0002	0023	0018	
4.8	0002	.0001	.0001	0013	0010	
4.9	0001	.0000	.0000	0009	0006	
5.0	.0000.	0000.	.0000	0006	0003	3333
5.1	0000.	0000.	0000	0004	0001	
5.2	0000.	0000.	.0000	0002	.0000	
5.3	0000.	0000.	.0000	.0001	.0000	

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TABLE VI. - ASYMPTOTIC VALUES

APPEARING IN EXPRESSION FOR

DISPLACEMENT THICKNESS

	[Pr, 0.	72; Y, 1.4	A NACA
	N = 1	N = 2	N = 3
aNI	-4.4764	-6.5114	-7.9898
α _{NS}	.0126	1230	2688
α <mark>N</mark> 3	-5.1946	-8.3220	-10.6824
β _{Nl}	-2.0346	-3.0786	-3.8104
β_{N2}	.0566	.0096	0476
BNI	-1.8294	-2.9060	-3.6564
BN2	.0680	.0236	0348
^B N3	7794	-2.3438	-3.8602
$B_{\rm N4}$	-16.0112	-23.4228	-29.1598
B _{N5}	-13.8116	-24.7052	-33.6536

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TABLE VII. - SOLUTION OF FIRST ORDER MOMENTUM EQUATION

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[Pr, 1; γ, 1.40]

η	g ₁₂	g ₁₂	g"	η	в ₁₂	s ¦2	g"12	N
0 1.2.3.4	0.0000 .0016 .0060 .0123 .0198	0.0000 .0312 .0544 .0701 .0787	0.3516 .2719 .1941 .1203 .0521	2-5 2.6 2.7 2.8 2.9	0.0222 .0209 .0199 .0191 .0136	-0.0150 0115 0087 0063 0045	0.0374 .0318 .0260 .0206 .0158	1 1 1 1
.5 .6 .7 .8 .9	.0278 .0357 .0431 .0494 .0545	.0808 .0775 .0692 .0375 .0434	0085 0599 1010 1308 1469	3.0 3.1 3.2 3.3 3.4	.0182 .0180 .0178 .0177 .0177	0032 0022 0014 0009 0006	.0117 .0085 .0060 .0041 .0027	1 1 1 1
1.0 1.1 1.2 1.3 1.4	.0581 .0601 .0606 .0598 .0578	.0281 .0126 0019 0143 0254	1555 1516 1385 1183 0933	3.5 3.6 3.7 3.8 3.9	.0175 .0175 .0175 .0175 .0175	0004 0002 0001 0001 .0000	.0 0 1 8 .0 0 1 1. .0 0 0 7 .0 0 0 4 .0 0 0 3	1 1 1 1
1.5 1.6 1.7 1.8 1.9	.0548 .0512 .0472 .0430 .0389	0334 0356 041% 0414 0398	0660 0387 0135 .0080 .0249	4.0 4.1 4.2 4.3 4.4	.0175 .0175 .0175 .0175 .0175	.0000 0000 .0000 0000 .0000	$\begin{array}{c} .0 & 0 & 0 & 2 \\ .0 & 0 & 0 & 1 \\ .0 & 0 & 0 & 0 \\ .0 & 0 & 0 & 0 \\ .0 & 0 & 0 & 0 \end{array}$	1 1 1 1
2.0 2.1 2.2 2.3 2.4	.0351 .0316 .0286 .0260 .0239	0366 0326 0280 0234 0190	.0367 .0437 .0454 .0456 .0423	4.5	.0175	.0000	.0000	1 1 1 1

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NACA IN 5028

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1.5 .10 1.0 .05 H_{N3} 2 •5 H_{N1} ${}^{\rm H}{}_{\rm N2}$ 0 .05 2 ŋ (a) H_{NL}. 0 õ 3 0 1 2 1 3 3 l η (Ъ) Η_{N2}. η (c) H_{N3}. 15 10 10 H_{N4} 5 H_{N5} 5 0 0 2 1 4 2 η η (d) H_{N4}. (e) H_{N5}. NACA

× 8

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Figure 4. - Airfoil shapes used in examples.







Figure 6. - Local skin friction as a function of distance from leading edge. $M_{\rm m}$, 3.



(b) Favorable pressure gradient.

Figure 7. - Average friction drag as a function of distance from leading edge. $M_{\rm w}$, 3.

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Figure 11. - Velocity profiles. M_{∞} , 3; x, 1.





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(b) Favorable pressure gradient.





Figure 14. - Comparison of present results with results of reference 13. Zero heat transfer; adverse pressure gradient; M_{∞} , 3.

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