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TECHNICAL NOTE 3401

LAMINAR BOUNDARY LAYER BEHIND SHOCK ADVANCING INTO STATIONARY FLUID

By Harold Mirels

SUMMARY

A study was made of the laminar compressible boundary layer induced by a shock wave advancing into a stationary fluid bounded by a wall. For weak shock waves, the boundary layer is identical with that which occurs when an infinite wall is impulsively set into uniform motion (Rayleigh problem). A numerical solution was required for strong shocks.

Velocity and temperature profiles, recovery factors, and skinfriction and heat-transfer coefficients are tabulated for a wide range of shock strengths.

INTRODUCTION

If a shock wave advances into a stationary fluid bounded by a wall, a boundary-layer flow is established along the wall behind the shock. This boundary layer is often important in studies of phenomena involving nonstationary shock waves. In a shock tube, for example, this boundary layer acts to attenuate the strength of the shock which propagates through the low-pressure side of the tube (refs. 1 and 2). If the shock tube is used as an aerodynamic wind tunnel, the test time available may depend, for long shock tubes, on the time it takes the boundary layer to introduce nonuniformities in the test section.

Another example of a shock-generated boundary layer occurs when a combustible mixture is ignited within a tube. In this case, a shock wave, followed by a flame front, is observed, as discussed in references 3 to 5. The shock wave is particularly strong when ignition occurs at a closed end. For long tubes, the progress of the flame front will be related to the boundary-layer development behind the shock. Since flame speed is increased by fluid turbulence, a transition from laminar to turbulent flow will accelerate the flame. Thus, the boundary layer may play a role in the acceleration of a low-speed flame to a detonation wave in a long tube (ref. 4).

The boundary layer behind the shock was studied in references 1 and 2. Both papers were primarily concerned with shock-wave attenuation. In reference 1, an approximate solution for the boundary layer was estimated by referring to the flow induced within a circular cylinder that is impulsively set into uniform translation. The validity of this analogy was not established. In reference 2, the correct boundary-layer equations were considered. These were integrated with a REAC (Reeves Electronic Analog Computer). Values of skin-friction and heat-transfer coefficients were presented. However, no velocity or temperature profiles were reported. Because of the growing interest in phenomena related to these shock-induced boundary layers, it was felt that a more detailed and more accurate study of this boundary-layer problem was warranted. Such a study was conducted at the NACA Lewis laboratory and the results are presented herein.¹

In the following sections, the laminar compressible boundary layer behind a shock wave advancing into a stationary fluid, bounded by a wall, is analyzed. For weak shocks, an analytical perturbation solution is presented. Numerical results for velocity and temperature profiles and heattransfer and skin-friction coefficients are tabulated, covering the range from weak to strong shocks. The numerical results are correct to four decimal places.

ANALYSIS

Coordinate Systems

A shock wave of constant strength is considered to move, parallel to a wall, into a stationary fluid. Let $(\overline{x}, \overline{y})$ be a coordinate system fixed in respect to the wall and let \overline{u} and \overline{v} be velocities parallel to the \overline{x} and \overline{y} coordinates, respectively, as indicated in figure 1(a). The flow is unsteady in this coordinate system. Let (x,\overline{y}) represent a coordinate system moving with the shock wave (fig. 1(b)). The velocities parallel to the x and y coordinates are denoted by u and v. In this coordinate system, the flow is steady.

Assume that, at time t = 0, the two coordinate systems coincide. If \overline{u}_s is the velocity of the shock wave relative to the wall, then \overline{x} and x are related by $x = \overline{x} - \overline{u}_s t$. Similarly, the axial velocities are related by $u = \overline{u} - \overline{u}_s$. Note that the wall moves with velocity $u_w = -\overline{u}_s$ in the steady coordinate system.

¹The writer's present interest in the shock-induced boundary-layer problem was stimulated by a private communication from Prof. N. Rott of Cornell University, who is studying heat-transfer problems associated with shock tubes.

Boundary-Layer Equations

The Prandtl boundary-layer equations apply for the flow in the vicinity of the wall (except at the base of the shock, where the boundary-layer assumptions break down). Assuming the flow to be laminar and dp/dx = 0, the equations of motion are, for x > 0,

$$\frac{\partial \rho u}{\partial x} + \frac{\partial \rho v}{\partial y} = 0 \text{ (continuity)}$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \frac{1}{\rho} \frac{\partial}{\partial y} \left(\mu \frac{\partial u}{\partial y}\right) \text{ (momentum)}$$

$$\rho c_p \left(u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y}\right) = \frac{\partial}{\partial y} \left(k \frac{\partial T}{\partial y}\right) + \mu \left(\frac{\partial u}{\partial y}\right)^2 \text{ (energy)}$$

$$p = \rho RT \text{ (state)}$$
(1)

The additional symbols are defined in appendix A. The boundary conditions for x > 0 are

$$u(x,0) = -\overline{u}_{g} \qquad u(x,\bullet) = u_{e}$$

$$v(x,0) = 0 \qquad T(x,\infty) = T_{e}$$

$$T(x,0) = T_{w}$$
(2)

These are the usual boundary conditions, except that the fluid at the wall moves with velocity $u(x,0) \equiv u_w = -u_g$ in order to satisfy the condition of zero slip at the wall. It will be assumed that the wall temperature T_w is constant. The magnitudes of u_e , T_e , and u_g depend on the shock strength and can be found from the normal shock relations quoted in appendix B. The ratio u_w/u_e increases from a value of 1 for a very weak shock to a value of $(\gamma + 1)/(\gamma - 1)$ for a very strong shock wave. Thus, in the steady coordinate system, the u velocities in the boundary layer have a maximum at the wall and decrease monotonically to the value in the free stream (as indicated in fig. 1(b)).

<u>Transformation</u>. - Equations (1) and (2) are transformed to a system of ordinary differential equations with the methods of references 6 and 7.

From the continuity equation, a stream function ψ exists such that $\partial \psi/\partial y = \rho u/\rho_w$, $-\partial \psi/\partial x = \rho v/\rho_w$. Following reference 6, a similarity parameter η is defined according to the relation

$$\eta = \sqrt{\frac{1}{2} \frac{u_e}{x v_w}} \int_0^y \frac{T_w}{T} dy$$
 (3)

and the stream function is written as

$$\Psi = \sqrt{2u_e x v_w} f(\eta)$$
 (4)

Note that $f' = u/u_e$. As in reference 7, the viscosity is assumed to vary linearly with temperature. If the viscosity is referenced to the wall value,

$$\mu = \left(\frac{\mu_{W}}{T_{W}}\right) T \tag{5}$$

Substitution of equations (3), (4), and (5) into the momentum equation yields

$$f^{111} + ff^{11} = 0$$
 (6)

with the boundary conditions

$$f(0) = 0$$

$$f'(0) = u_{W}/u_{e}$$

$$f'(\omega) = 1$$
(7)

Equation (6) is the familiar Blasius differential equation. However, the tangential velocity boundary condition at the wall $(f'(0) = u_w/u_e)$ is different from the zero value usually encountered in studies of viscous flow past a semi-infinite plate. Numerical integration is required except for the limiting case of a weak shock, $[(u_w/u_e) - 1] \ll 1$, for which an analytical perturbation solution is possible.

For $\ensuremath{\mathbb{T}}$ a function of $\ensuremath{\eta}$ only, the energy equation becomes

$$\mathbb{T}'' + \sigma f \mathbb{T}' = -\sigma(\gamma - 1) M_e^2(f'')^2 \tag{8}$$

assuming that the Prandtl number σ is constant. Since equation (8) is linear, the general solution for T can be expressed as the linear superposition of the solution for zero heat transfer plus the effect of heat transfer. That is, T/T_e can be expressed in the form

$$\frac{T}{T_e} = 1 + \frac{\gamma - 1}{2} \left[\left(\frac{u_w}{u_e} - 1 \right) M_e \right]^2 r(\eta) + \left\{ \left(\frac{T_w}{T_e} - 1 \right) - \frac{\gamma - 1}{2} \left[\left(\frac{u_w}{u_e} - 1 \right) M_e \right]^2 r(0) \right\} s(\eta)$$
(9)

where r(n) satisfies

$$\mathbf{r}^{"} + \sigma \mathbf{f}\mathbf{r}^{*} = \frac{-2\sigma}{\left(\frac{u_{W}}{u_{e}} - 1\right)^{2}} \left\{ \mathbf{f}^{"}\right\}^{2}$$

$$\mathbf{r}^{(\mathbf{w})} = \mathbf{r}^{*}(0) = 0$$
(10)

and $s(\eta)$ satisfies

$$s'' + \sigma fs' = 0$$

$$s(0) = 1; s(\infty) = 0$$
(11)

Note that $[(u_w/u_e) - 1] M_e \equiv |\overline{u_e}|/a_e$ is the Mach number of the external flow relative to the wall. For an insulated wall, the coefficient of s (in eq. (9)) equals zero, so that the wall temperature is

$$\frac{T_{w,i}}{T_e} = 1 + \frac{\gamma - 1}{2} \left[\left(\frac{u_w}{u_e} - 1 \right) M_e \right]^2 r(0)$$
 (12)

Thus, r(0) is a recovery factor based on the Mach number of the external flow relative to the wall. Equation (9), in terms of $T_{w,i}$, is

$$\frac{\underline{T}}{\underline{T}_{e}} = 1 + \frac{\underline{\gamma} - 1}{2} \left[\left(\frac{\underline{u}_{w}}{\underline{u}_{e}} - 1 \right) M_{e} \right]^{2} r(\eta) + \left(\frac{\underline{T}_{w}}{\underline{T}_{e}} - \frac{\underline{T}_{w,1}}{\underline{T}_{e}} \right) s(\eta) \qquad (12a)$$

Equations (10) and (11) can be expressed in quadrature form

$$\mathbf{r} = \frac{2\sigma}{\left(\frac{u_{w}}{u_{e}} - 1\right)^{2}} \int_{\eta}^{\bullet} \left[\mathbf{f}''(\boldsymbol{\xi})\right]^{\sigma} d\boldsymbol{\xi} \int_{0}^{\boldsymbol{\xi}} \left[\mathbf{f}''(\boldsymbol{\theta})\right]^{2-\sigma} d\boldsymbol{\theta}$$
(13)

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$$s = \int_{\eta}^{\infty} [f''(\xi)]^{\sigma} d\xi / \int_{0}^{\infty} [f''(\xi)]^{\sigma} d\xi$$
 (14)

For $\sigma = 1$, these equations can be integrated to yield

$$r = 1 - \left(\frac{\frac{u_{w}}{u_{e}} - f^{\dagger}}{\frac{u_{w}}{u_{e}} - 1}\right)^{2}$$

$$s = \frac{f^{\dagger} - 1}{\frac{u_{w}}{u_{e}} - 1}$$
(15)
(16)

The solutions for other Prandtl numbers are discussed later.

An alternate system of equations is described in appendix C.

Relation between y and $\eta.$ - For x constant, the relation between y and η is, from equation (3),

$$y \sqrt{\frac{1}{2} \frac{u_e}{x v_w}} = \int_0^{\eta} \frac{T}{T_w} d\eta$$
 (17)

Substitution of equation (12a) into equation (17) yields

$$y \sqrt{\frac{1}{2} \frac{u_e}{x v_w}} = \frac{T_e}{T_w} \left\{ \eta + \frac{\gamma - 1}{2} \left[\left(\frac{u_w}{u_e} - 1 \right) M_e \right]^2 \int_0^{\eta} r \, d\eta + \frac{T_w - T_w, 1}{T_e} \int_0^{\eta} s \, d\eta \right\}$$
(18)

For $\sigma = 1$, equations (15) and (16) can be substituted into equation (18) with the following result:

$$y \sqrt{\frac{1}{2} \frac{u_{e}}{xv_{w}}} = \frac{T_{e}}{T_{w}} \left\{ \eta + \frac{\frac{T_{w}}{T_{e}} - 1}{\frac{u_{w}}{u_{e}} - 1} (f - \eta) + \frac{\gamma - 1}{2} M_{e}^{2} \left[\frac{u_{w}}{u_{e}} (f - \eta) + f(1 - f') + f''(0) - f'' \right] \right\}$$
(19)

If the wall is insulated, equation (19) becomes

$$y \sqrt{\frac{1}{2} \frac{u_{e}}{xv_{w}}} = \frac{\eta + \frac{\gamma - 1}{2} M_{e}^{2} \left[(f - \eta) \left(\frac{2u_{w}}{u_{e}} - 1 \right) + f(1 - f') + f''(0) - f'' \right]}{1 + \frac{\gamma - 1}{2} \left[\left(\frac{u_{w}}{u_{e}} - 1 \right) M_{e} \right]^{2}}$$

$$(20)$$

Equations (18) to (20) are useful for obtaining velocity and temperature profiles in terms of y rather than η .

Perturbation Solution for
$$\left(\frac{u}{u_e} - 1\right) \ll 1$$

If the shock wave is weak, $(u_w/u_e) - 1$ is small, and a perturbation analysis in terms of this parameter is possible. Let

$$f = \sum_{n=0}^{\infty} f_n \left(\frac{u_w}{u_e} - 1 \right)^n$$
 (21a)

$$r = \sum_{n=0}^{\infty} r_n \left(\frac{u_w}{u_e} - 1 \right)^n$$
 (21b)

$$s = \sum_{n=0}^{\infty} s_n \left(\frac{u_w}{u_e} - 1\right)^n$$
 (21c)

Substituting equation (21a) into equations (6) and (7) and equating coefficients of $[(u_w/u_e) - 1]^0$ and $[(u_w/u_e) - 1]^1$ yield, respectively,

$$\begin{cases} f_0''' + f_0 f_0'' = 0 \\ f_0(0) = 0; f_0'(0) = f_0'(-) = 1 \end{cases}$$
(22)

$$\begin{array}{c} f_{1}''' + f_{0}f_{1}'' + f_{1}f_{0}'' = 0 \\ f_{1}(0) = 0; f_{1}'(0) = 1; f_{1}'(\bullet) = 0 \end{array}$$
 (23)

Integration of equations (22) and (23) gives

$$f_{0} = \eta$$

$$f_{1} = \eta \operatorname{erfc}\left(\frac{\eta}{\sqrt{2}}\right) + \sqrt{\frac{2}{\pi}}\left[1 - \exp\left(-\frac{\eta^{2}}{2}\right)\right]$$
(24)

so that

$$f = \eta + \left\{ \eta \operatorname{erfc}\left(\frac{\eta}{\sqrt{2}}\right) + \sqrt{\frac{2}{\pi}} \left[1 - \exp\left(-\frac{\eta^2}{2}\right) \right] \right\} \left(\frac{u_w}{u_e} - 1\right) + 0\left(\frac{u_w}{u_e} - 1\right)^2 \right\}$$

$$f' = 1 + \left[\operatorname{erfc}\left(\frac{\eta}{\sqrt{2}}\right) \right] \left(\frac{u_w}{u_e} - 1\right) + 0\left(\frac{u_w}{u_e} - 1\right)^2 + 0\left(\frac{u_w}{u_e} - 1\right)^2 \right\} \left(25\right)$$

$$f'' = -\sqrt{\frac{2}{\pi}} \left(\frac{u_w}{u_e} - 1\right) e^{-\eta^2/2} + 0\left(\frac{u_w}{u_e} - 1\right)^2 + 0\left(\frac{u_w}{u_e} - 1\right)^2$$

Substituting the preceding solution for f into equations (10) and (11) and equating coefficients of $\left(\frac{u_w}{u_e} - 1\right)^0$ yield

$$\begin{array}{c} r_{0}^{"} + \sigma \eta r_{0}^{i} = -\frac{4}{\pi} \quad \sigma \ e^{-\eta^{2}} \\ r_{0}^{i}(0) = r_{0}(\bullet) = 0 \\ s_{0}^{"} + \sigma \eta s_{0}^{i} = 0 \\ s_{0}(0) = 1; \ s_{0}(\bullet) = 0 \end{array} \right\}$$

$$(26)$$

$$(27)$$

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Integration of equation (26) yields (from results of ref. 8)

$$r = \frac{4}{\pi} \left(\frac{\sigma}{2-\sigma}\right)^{1/2} \int_{\sin^{-1}\sqrt{\frac{\sigma}{2}}}^{\pi/2} \exp\left(\frac{-\sigma\eta^2}{2}\frac{1}{\sin^2\theta}\right) d\theta + O\left(\frac{u_w}{u_e} - 1\right)$$
(28)

The integral of equation (27) is

$$s = \operatorname{erfc}\left(\sqrt{\frac{\sigma}{2}} \eta\right) + O\left(\frac{u_{w}}{u_{e}} - 1\right)$$
(29)

From equations (28) and (29),

$$\mathbf{r}(0) = \frac{4}{\pi} \left(\frac{\sigma}{2-\sigma}\right)^{1/2} \left(\frac{\pi}{2} - \sin^{-1} \sqrt{\frac{\sigma}{2}}\right) + 0 \left(\frac{u_{w}}{u_{e}} - 1\right)$$
(30)

$$\mathbf{s}'(\mathbf{0}) = -\sqrt{\frac{2\sigma}{\pi}} + O\left(\frac{u_w}{u_e} - 1\right)$$
(31)

where r(0) is the recovery factor and s'(0) is used to calculate heat transfer. Equations (22) to (31) (neglecting the higher-order terms) apply if an infinite plate is started, impulsively, with velocity $[(u_w/u_e) - 1]$. The latter, often termed the "Rayleigh problem," has been much discussed. The zero-order solution indicated by equations (28) and (30) was obtained in reference 8 in a study of the aerodynamic heating associated with the Rayleigh problem.

In reference 1, the boundary layer behind a shock wave advancing into a stationary fluid was estimated by analogy with the Rayleigh problem. The work of the present section shows that this approach is exact for weak shocks.

Numerical Solution

For other than weak shock waves, a numerical solution of equations (6), (10), and (11) is required. This was obtained by Lynn U. Albers with the use of an IEM card-programmed electronic calculator. The integration technique is described in appendix B of reference 9. The results, correct to four decimal places, are tabulated in table I. Values of f, f', and f" are given for u_w/u_p of 1.5, 2, 3, 4, 5, and 6, while values of r,

r', s, s', $\int_0^{\eta} r \, d\eta$, and $\int_0^{\eta} s \, d\eta$ are presented for u_w/u_e of 2, 4, and 6 (with $\sigma = 0.72$). The data of table I plus the perturbation solution of the previous section define solutions covering the range from very weak shocks to very strong shocks.

RESULTS AND DISCUSSION

In the steady coordinate system, the boundary-layer similarity parameter is $\eta = \sqrt{\frac{1}{2} \frac{u_e}{v_w x}} \int_0^y \frac{T_w}{T} dy$. Transforming to the unsteady coordinate system according to the relation $x = \overline{x} + u_w t$ and considering station $\overline{x} = 0$ gives $\eta = \sqrt{\frac{1}{v_w t}} \sqrt{\frac{u_e}{2u_w}} \int_0^y \frac{T_w}{T} dy$. Thus, the boundary layer behind the shock wave has features of a Blasius type flow or a Rayleigh type flow, depending on whether the observer is stationary with respect to the shock wave, or wall, respectively.

The Rayleigh viewpoint is used herein to correlate and discuss the numerical data. That is, attention is fixed at station $\bar{x} = 0$, and the boundary-layer development for t > 0 is considered. The velocity which characterizes the boundary-layer development is $u_w - u_e$. Similarly, a characteristic length is $(u_w - u_e)$ t, which is the distance a particle in the free stream moves relative to the wall in time t. The form of the Reynolds number used herein can then be defined as

$$Re \equiv \frac{(u_w - u_e)^2 t}{v_w}$$
(32)

Boundary-Layer Profiles

The parameter $[(u_w/u_e) - f']/[(u_w/u_e) - 1]$ varies from a value of zero at the wall to a value of 1 at the edge of the boundary layer. A boundary-layer thickness δ may be defined as the value of y corresponding to $[(u_w/u_e) - f']/[(u_w/u_e) - 1] = 0.99$. Values of the boundary-layerthickness parameter $\delta/\sqrt{v_e t}$ have been computed assuming $\gamma = 1.4$ and an insulated plate, for σ of 0.72 and 1.0 (using eqs. (18) and (20)). These are tabulated in table II. It is seen that $(\delta/\sqrt{v_e t})_i$ increases with increasing u_w/u_e . As expected, the values for $\sigma = 1$ are greater than those for $\sigma = 0.72$ (for $u_w/u_e \neq 1$). This is due to the fact NACA TN 3401

that the larger recovery factor of the former leads to higher temperatures near the wall and therefore a larger value of y corresponding to a given η .

Velocity profiles for u_w/u_e of 1 (limiting case of very weak shocks) and 6 are plotted in figure 2. Curves for intermediate values of u_w/u_e lie smoothly between the curves in the figure. No marked departure from an error function type velocity profile is indicated.

Skin Friction and Heat Transfer

The shear stress at the wall is given by

$$\tau_{\rm w} = \mu_{\rm w} \left(\frac{\partial u}{\partial y} \right)_{\rm w} = \mu_{\rm w} u_{\rm e} \sqrt{\frac{1}{2} \frac{u_{\rm e}}{x v_{\rm w}}} f''(0)$$

Because of the coordinate system used, τ_w is negative. If $(u_w - u_e)$ is used as a reference velocity, a positive local friction coefficient can be defined as $c_f = -\tau_w / \frac{1}{2} \rho_w (u_w - u_e)^2$. Then, using the Reynolds number as defined by equation (32),

$$c_{f} \sqrt{Re} = \frac{-\sqrt{2} f''(0)}{\sqrt{\frac{u_{w}}{u_{e}}} \left(\frac{u_{w}}{u_{e}} - 1\right)}$$
(33)

Values of $c_f \sqrt{Re}$ are tabulated in table II. These vary from the Rayleigh value of 1.128 at $u_w/u_e = 1$ to 0.935 at $u_w/u_e = 6$. The corresponding value of $c_f \sqrt{Re}$ for incompressible flow past a semi-infinite plate (Blasius problem) is 0.664.

The heat transferred into the fluid from a unit area of the wall, per unit time, is

$$q = -k_{w} \left(\frac{\partial T}{\partial y} \right)_{w} = -k_{w} \sqrt{\frac{1}{2} \frac{u_{e}}{x v_{w}}} \left(T_{w} - T_{w,1} \right) s'(0)$$
(34)

Defining a heat-transfer coefficient h by $h = (T_w - T_{w,i})/q$ and a Nusselt number by the relation $Nu = h(u_w - u_e) t/k_w$ permits the Nusselt number to be written as

$$Nu = \frac{-s'(O)}{\sqrt{\frac{1}{2} \frac{u_w}{u_e}}} \sqrt{Re}$$

The relation between skin friction and heat transfer can be expressed in terms of a Reynolds analogy parameter $\sqrt{\sigma} c_f \text{Re/Nu}$. The factor $\sqrt{\sigma}$ is included, since, for $u_w/u_e = 1$, $\sqrt{\sigma} c_f \text{Re/Nu}$ is a constant. For $\sigma = 1$, the parameter equals 2 at all values of u_w/u_e ; while for $\sigma = 0.72$, the parameter increases from 2 at $u_w/u_e = 1$ to 2.07 at $u_w/u_e = 6$.

The recovery factor for $u_w/u_e = 1$ is given by equation (30). Evaluating this equation for $\sigma = 0.72$ gives r(0) = 0.885. The recovery factor for $\sigma = 0.72$ increases with increasing u_w/u_e to a value of 0.920 at $u_w/u_e = 6$. These compare with the value 0.845 for flow past a semiinfinite plate at $\sigma = 0.72$.

Thus, for the range of u_w/u_e investigated, the numerical results for skin friction, heat transfer, and recovery factor (in terms of the parameters defined herein) depart relatively less from Rayleigh $(u_w/u_e = 1)$ values than from the Blasius values for equivalent flows past a semiinfinite flat plate.

CONCLUDING REMARKS

The laminar boundary layer behind a shock wave advancing into a stationary fluid, bounded by a wall, has been determined. Various boundarylayer parameters have been tabulated for several shock strengths.

With increasing Reynolds numbers, the laminar boundary layer behind the shock will become unstable, and transition to turbulent flow will ultimately occur. A theoretical study of the stability of this laminar boundary layer would be of interest. Shock-tube experiments might provide criteria defining the transition to turbulent boundary-layer flow as well as the characteristics of the turbulent boundary layer. At present, little is known about the structure of such turbulent boundary layers, and it is felt that some experimental data should be available before an analytical study is attempted.

Lewis Flight Propulsion Laboratory National Advisory Committee for Aeronautics Cleveland, Ohio, December 10, 1954 .

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APPENDIX A

SYMBOLS

The following symbols are used in this report:

| a | speed of sound |
|----------------|--|
| c _f | local skin-friction coefficient, $-\tau_w / \left[\frac{1}{2} \rho_w (u_w - u_e)^2\right]$ |
| с _р | specific heat at constant pressure |
| erf(x) | error function, $(2/\sqrt{\pi}) \int_0^x e^{-y^2} dy$ |
| erfc(x) | complementary error function, 1 - erf(x) |
| f | function of η defined by eq. (4) |
| h | heat-transfer coefficient, $(T_w - T_{w,i})/q$ |
| k | thermal conductivity |
| м | Mach number |
| Me | u _e /a _e |
| Nu | Nusselt number, $h(u_w - u_e)t/k_w$ |
| р | pressure |
| đ | local rate of heat transfer |
| R | gas constant |
| Re | Reynolds number, $(u_w - u_e)^2 t / v_w$ |
| r | function defined by eq. (10) |
| r(0) | recovery factor |
| 8 | function defined by eq. (11) |
| Т | static temperature (abs) |
| t | time |
| u,v | velocities parallel to x and y coordinates, respectively |

us velocity of shock wave relative to wall

- x,y coordinates parallel to and normal to wall, respectively, and moving with shock wave (fig. l(b))
- $\overline{x}, \overline{y}$ coordinates parallel to and normal to wall, respectively, and stationary with respect to wall (fig. l(a))
- γ ratio of specific heats
- δ value of y for which $(u_w/u_e u/u_e)/(u_w/u_e 1) = 0.99$ (i.e., boundary-layer thickness)
- η variable defined by eq. (3)
- η_{δ} value of η at $y = \delta$
- μ coefficient of viscosity
- v kinematic viscosity

ρ mass density

- σ Prandtl number, $\mu c_{0}/k$
- ψ stream function

Subscripts:

| Ъ | undisturbed flow in front of shock wave (appendix B) |
|---|--|
| e | flow external to boundary layer |
| i | insulated-wall problem |
| w | conditions at wall $(\mathbf{v} \neq \overline{\mathbf{y}} = 0)$ |

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APPENDIX B

SHOCK RELATIONS

For convenience, some useful shock-wave relations are noted herein:



Consider flow in a steady coordinate system. Let subscript b designate undisturbed flow in front of the shock wave; and subscript e, the flow behind the shock wave and external to the boundary layer. Note that $u_b = u_w$, so that $M_b \equiv u_b/a_b = u_w/a_b$. Then,

$$\frac{u_{w}}{u_{e}} = \frac{(\gamma+1)M_{b}^{2}}{(\gamma-1)M_{b}^{2}+2}$$

$$= \frac{6M_{b}^{2}}{M_{b}^{2}+5} \quad \text{for } \gamma = 1.4$$
(B1)

$$M_{e}^{2} = \frac{2}{(\gamma+1) \frac{u_{w}}{u_{e}} - (\gamma-1)}$$

$$= \frac{5}{6 \frac{u_{w}}{u_{e}} - 1} \quad \text{for } \gamma = 1.4$$
(B2)

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APPENDIX C

ALTERNATE FORMULATION OF BOUNDARY-LAYER PROBLEM

The transformation based on equation (3) leads to a system of equations identical to that which arises in boundary-layer studies of the flow past a semi-infinite flat plate, except for the non-zero velocity boundary condition at the wall. This sytem was convenient for numerical computation, since a variety of flat-plate boundary-layer problems had previously been solved at the NACA Lewis laboratory, and the cardprogramming for the IBM electronic calculator was already established.

An alternate system can be obtained by normalizing the momentumequation boundary conditions. That is, the parameter u_w/u_e appears in the differential equation rather than in the boundary conditions. Such a system leads more directly to the correct physical parameters of the problem and is described by the following equations.

Define ζ and $g(\zeta)$ according to the relations

$$\zeta \equiv \sqrt{\frac{u_w}{u_e}} \eta \tag{C1}$$

$$g(\zeta) = \frac{1}{\frac{u_w}{u_e} - 1} \left(\frac{u_w}{u_e} \zeta - \sqrt{\frac{u_w}{u_e}} f \right)$$
(C2)

Using equations (C1) and (C2), equations (6), (7), (10), and (11) become

$$\frac{d^{3}g}{d\xi^{3}} + \frac{d^{2}g}{d\xi^{2}} \left(\zeta - \frac{\frac{u_{w}}{u_{e}} - 1}{\frac{u_{w}}{u_{e}}} g \right) = 0$$

$$g(0) = \left(\frac{dg}{d\xi} \right)_{\xi=0} = 0; \quad \left(\frac{dg}{d\xi} \right)_{\xi=0} = 1$$

$$\frac{d^{2}r}{d\xi^{2}} + \sigma \left(\zeta - \frac{\frac{u_{w}}{u_{e}} - 1}{\frac{u_{w}}{u_{e}}} g \right) \frac{dr}{d\xi} = -2\sigma \left(\frac{u_{w}}{u_{e}} - 1 \right) \left(\frac{d^{2}g}{d\xi^{2}} \right)^{2}$$

$$r(\bullet) = \left(\frac{dr}{d\xi} \right)_{\xi=0} = 0$$

$$(C3)$$

$$(C4)$$

$$\frac{d^{2}s}{d\zeta^{2}} + \sigma \left(\zeta - \frac{\frac{u_{W}}{u_{e}} - 1}{\frac{u_{W}}{u_{e}}} g \right) \frac{ds}{d\zeta} = 0$$

$$s(0) = 1; \quad s(\infty) = 0$$
(C5)

Note that, for $\bar{\mathbf{x}} = 0$, $\zeta = 1/\sqrt{2\nu_w t}$, which is the correct parameter for the Rayleigh problem. Also $dg/d\zeta = [(u_w/u_e) - f^*]/[(u_w/u_e) - 1]$, which is a normalized velocity. Finally, the reduction of the system to a Rayleigh problem, for $u_w/u_e = 1$, is more apparent from equations (C3) to (C5) than from equations (6) to (11).

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| | u _w /u _g = 1.5 | | | [| u _w /u _e = 3.0 | | u _w /u _e = 5.0 | | |
|---------------------------------|---|--|---|--|---|---|---|--------------------------------------|---|
| 71 | r | f' | -f" | f | f | -1" | ſ | f¹ | -f ⁿ |
| 0.0 | 0,0000 | 1.5000 | 0.4 578 | 00000 | 3.0000 | 23973 | 0.0000 ⁻ | 50000 | 5,9726 |
| .1 | 1477 | 1.4543 | .4 5 4 4 | .2880 | 8.7614 | 23620 | ,4703 | 44075 | 5,8309 |
| .2 | 2909 | 1.4093 | .4 4 4 5 | .5525 | 2.5296 | 22649 | ,8824 | 38419 | 5,4470 |
| .3 | ,4296 | 1.3656 | .4 2 8 8 | .7944 | 3.3101 | 21170 | 1.8402 | 33837 | 4,8964 |
| .4 | ,5641 | 1.3238 | .4 0 8 0 | 10151 | 21074 | 19336 | 1.5492 | 28656 | 4,2573 |
| .5 | .6945 | 1,2842 | .3831 | 1.2165 | 19241 | 1,7291 | 1,8155 | 24729 | 35968 |
| .6 | .8210 | 18472 | .3551 | 1.4005 | 17618 | 1,5168 | 2,0459 | 21452 | 29644 |
| .7 | .9440 | 1,2158 | .3851 | 1.5695 | 16807 | 1,3073 | 2,2456 | 18780 | 23913 |
| .8 | 10637 | 1,1828 | .2941 | 1.7354 | 15000 | 1,1087 | 2,4233 | 16645 | 18930 |
| .9 | 11606 | 1,1544 | .2628 | 1.8703 | 13984 | .9261 | 2,5810 | 14968 | 14737 |
| 1.0 | 1.8947 | 1.1297 | .2322 | 8.0057 | 13141 | ,7629 | 8.7839 | 13672 | 11303 |
| 1.1 | 1.4086 | 1.1079 | .2029 | 8.1355 | 12451 | ,6203 | 2.8555 | 12685 | .8551 |
| 1.2 | 1.5164 | 1.0890 | .1753 | 2.2551 | 11894 | ,4980 | 2.9784 | 11943 | .6387 |
| 1.3 | 1.6845 | 1.0728 | .1498 | 2.3718 | 11449 | ,3952 | 3.0950 | 11391 | .4714 |
| 1.4 | 1.7310 | 1.0590 | .1267 | 8.4844 | 11097 | ,3100 | 3.8067 | 10986 | .3440 |
| 1.5 | 1,8363 | 1,0474 | 1060 | 2,5940 | 10823 | 2405 | 33151 | 10692 | .2483 |
| 1.6 | 1,9406 | 1,0377 | .0877 | 2,7011 | 10612 | 1845 | 34209 | 10481 | .1773 |
| 1.7 | 2,0439 | 1,0297 | .0719 | 2,8064 | 10451 | 1401 | 35249 | 10331 | .1253 |
| 1.8 | 2,1466 | 1,0233 | .0583 | 2,9102 | 10329 | 1053 | 36276 | 10226 | .0876 |
| 1.9 | 2,3486 | 1,0180 | .0468 | 3,0131 | 10237 | 0783 | 37295 | 10153 | .0606 |
| 30 | 2,3502 | 1.0138 | .0378 | 31151 | 10170 | .0576 | 16308 | 10103 | 0415 |
| 21 | 2,4514 | 1.0105 | .0292 | 32165 | 10120 | .0420 | 19316 | 10068 | 0282 |
| 23 | 2,5523 | 1.0079 | .0228 | 33175 | 10085 | .0303 | 40322 | 10044 | 0189 |
| 34 | 2,6530 | 1.0059 | .0176 | 34189 | 10059 | .0216 | 41325 | 10039 | 0126 |
| 24 | 2,7535 | 1.0044 | .0134 | 35187 | 10041 | .0153 | 42326 | 10019 | 0083 |
| 25 | 28539 | 1.0032 | .0101 | 36191 | 10028 | .0107 | 43389 | 1.00121.00081.00051.00031.00031.0008 | .0054 |
| 26 | 29542 | 1.0023 | .0076 | 37193 | 10019 | .0074 | 44330 | | .0035 |
| 27 | 30544 | 1.0017 | .0056 | 38195 | 10013 | .0051 | 45331 | | .0022 |
| 28 | 31545 | 1.0012 | .0041 | 39196 | 10008 | .0035 | 46331 | | .0014 |
| 29 | 32546 | 1.0008 | .0030 | 40196 | 10005 | .0035 | 47331 | | .0019 |
| 3.0 3.1 3.2 3.3 3.4 | 33547 34548 35548 36548 36548 37548 | 1.0006 1.0004 1.0003 1.0003 1.0003 | .0021 .0015 .0011 .0007 .0005 | 4.1 19 7 4.2 19 7 4.3 19 7 4.4 19 7 4.5 19 7 | 10004 10003 10001 10001 10001 | .0015 .0010 .0007 .0004 .0003 | 48331 49332 50332 51332 52332 | 1.00011.00011.00001.00001.00001.0000 | .0005 .0003 .0002 .0001 .0001 |
| 3,5 3,6 3,7 3,8 3,9 | 3,8 5 4 9 3,9 5 4 9 4,0 5 4 9 4,1 5 4 9 4,2 5 4 9 | 1.00011.00011.00001.00001.00001.0000 | .0004 .0002 .0008 .0001 .0001 | 4.6197 4.7197 4.8197 4.9197 | 10000 10000 10000 10000 | 0000 0001 0001 0000 | 5,3332 | 10000 | .0000 |
| 4,0 4,1 4,3 4,3 4,4 | 43549 | 1.0000 | .0000 | | | | | | |

TABLE I. - LANINAR BOUNDARY LAYER BEHIND SHOCK WAVE

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(a) Solution of momentum equation for $u_{\rm sp}/u_{\rm g}$ = 1.5, 3.0 and 5.0

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| PABLE I Continued. LAMINAR BOUNDARY LAYER EEHIND SHOCK WAVE | |
|--|--|
| (b) Solution of momentum and energy equation for $u_{\rm g}/u_{\rm g}=2.0$ | |
| | |

and the second second

| (0) | Solution | or | momentum | and | energy | equation | for | _u_/u | | 2.0 |) |
|-----|----------|----|----------|-----|--------|----------|-----|-------|--|-----|---|
|-----|----------|----|----------|-----|--------|----------|-----|-------|--|-----|---|

| | / · | ſ | | Prandtl number, $\sigma = 0.72$ | | | | | |
|---------------------------------|---|--|---|--|--|--|--|--|--|
| ח | f | t' | -£* | | r | ge 1 | $\int_0^{\eta} dn$ | L II | ~=' |
| 0,0 1,2 1,2 1,4 | 0-0000 1949 3798 .5548 .7205 | 2.0000 1.8984 1.7988 1.7029 1.6121 | 10191 10091 9804 9356 8777 | 0,0000 .0897 .1780 .2634 .3447 | 0.8997 .8992 .8704 .8356 .7898 | 0.0000 .1479 .2861 .4068 .5043 | 0.0000 .0958 .1831 .2621 .3331 | 1.0000 .9151 .8313 .7498 .6715 | 0.8512 .8452 .8279 .8004 .7645 |
| 5 5 7 9 9 | .8774 10263 11677 13026 14316 | 1.5276 1.4503 1.3804 1.3183 1.8637 | 8103 7367 5601 5834 5088 | .4210 .4916 .5560 .6141 .6657 | -7356 -6785 -6199 -5490 -4849 | .5758 .6209 .6412 .6398 .6206 | .3965 .4527 .5088 .5454 .5839 | .5978 .5374 .4625 .4089 .3486 | .7217 .6739 .6227 .5697 .8162 |
| 1.0 1.1 1.9 1.3 1.4 | 1.5556 1.6751 1.7910 1.9036 2.0137 | 1.2164 11759 11416 1.1139 1.0893 | .4382 3728 3135 2606 3143 | .7111 .7507 .7848 .8139 .8386 | .4244 .3676 .3135 .8683 .2262 | 5878 5455 4974 4465 3953 | .6153 .6430 .6666 .6866 .7034 | .2996 .2558 .2170 .1829 .1531 | .4636 .4127 .3643 .3189 .8769 |
| 1.5 1.6 1.7 1.9 | 21216 82878 33386 84363 25390 | 1.0699 1.0542 1.0417 1.0317 1.0239 | 1748 1402 1116 0879 0686 | .8594 .8766 .8909 .9036 .9123 | .1898 .1570 .1293 .1057 .0858 | .3456 .3988 .2556 .2166 .1819 | .7174 .7290 .7385 .7463 .7527 | .1274 .1053 .0864 .0705 .0572 | .2386 .8040 .1731 .1458 .1219 |
| 2,0 21 2,2 2,3 2,4 | 8.6411 2.7426 2.8436 2.9446 3.0453 | 1.0179 1.0133 1.0097 1.0071 1.0051 | ,0529 .0404 .0306 .0229 .0170 | .9199 .9261 .9310 .9350 .9361 | .0692 .0554 .0441 .0348 .0273 | .1514 .1250 .1024 .0832 .0671 | .7578 .7620 .7653 .7679 .7699 | .0460 .0368 .0293 .0231 .0231 | .1013 .0833 .0688 .0553 .0446 |
| 2,5 2,5 2,7 2,8 2,9 | 3,1 4 5 6 3,8 4 5 9 3,3 4 6 3 3,4 4 6 3 3,5 4 6 4 | 1.0036 1.0036 1.0018 1.0018 1.0018 | 0124 0090 0065 0046 0033 | .9405 .9424 .9438 .9449 .9458 | .0813 .0165 .0187 .0097 .0073 | 0538 0487 0337 0264 0205 | .7715 .7726 .7737 .7745 .7750 | .0141 .0109 .0064 .0064 | .0357 .0284 .0294 .0175 .0136 |
| 3.0 3.1 3.8 3.3 3.4 | 3.6465 3.7465 3.8466 3.9466 4.0466 | 1.00061.00041.00031.00031.00031.0001 | 0023 0016 0011 0007 0007 | .9464 .9469 .9473 .9475 .9477 | .0055 .0041 .0031 .0023 .0017 | .0158 .0181 .0098 .0070 .0058 | .7755 .7758 .7760 .7762 .7763 | .0037 .0088 .0080 .0015 .0011 | .0105 .0081 .0061 .0046 .0035 |
| 35 36 37 38 39 | 4.1 465 4.8 465 4.3 465 4.4 465 4.5 465 | 1.00011.00001.00001.00001.00001.0000 | 0003 0002 0001 0001 0001 | .9479 .9480 .9480 .9481 .9481 | .0012 .0009 .0006 .0005 .0003 | .0039 .0029 .0021 .0015 .0011 | .7764 .7765 .7765 .7766 .7766 .7766 | .0008 .0006 .0004 .0003 .0003 | .0096 .0019 .0014 .0010 .0007 |
| 40 41 43 43 44 | 4,6 4 6 6 | 1,0000 | ·00000 | .9482 .9482 .9482 .9482 .9482 | .0002 .0003 .0001 .0001 .0001 | .0008 .0004 .0003 .0003 | .7766 .7766 .7766 .7766 .7768 .7765 | .0001 .0001 .0001 .0001 .0001 | .0005 .0004 .0003 .0002 .0001 |
| 4.5 4.6 4.7 4.8 4.9 | | | | .9482 .9482 .9482 .9482 | 0000 0000 0000 0000 | .0001 .0001 .0001 .0001 | .7767 .7767 .7767 .7767 .7767 | 00000. 00000. 00000. 00000. | .0001 .0001 .0000 .0000 |

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TABLE I. - Continued. LANDMAR BOUNDARY LAYER HEHIND SHOCK WAVE

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. <u>5. 19</u>11.

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(c) Solution of momentum and energy equation for $u_{\rm g}/u_{\rm g}=4.0$

| | | ļ | ļ <u> </u> | Prendtl number, $\sigma = 0.72$ | | | | | |
|---------------------------------|---|--|---|--|--|--|--|--|---|
| η | Ţ | £1 | -1 - 1 | $\int_0^{\eta} r d\eta$ | , r | - <u>r</u> 1 | $\int_0^{\eta} a d\eta$ | | -8' |
| 0,0 .1 .8 .3 .4 | 00000 3798 7198 10222 12901 | 4.0000 3.5964 3.8077 2.8457 2.5188 | 40623 39845 37702 34546 30766 | 0.0000 .0909 .1791 .2626 .3396 | 0.9129 .8998 .8623 .8047 .7328 | 0.0000 .2582 .4847 .6579 .7690 | 0.0000 .0945 .1779 .2509 .3139 | 1.0000 .8898 .7811 .6784 .5830 | 11186 11005 10576 .9931 .9136 |
| .5 .6 .7 .9 | 1,5 273 1,7 377 1,9 355 2,0 9 4 3 3,2 4 7 6 | 22313 19844 17768 16055 14664 | 25717 23688 18867 15446 13430 | .4089 .4701 .5831 .5683 .6062 | .6529 .5705 .4900 .4145 .3461 | .8202 .8209 .7838 .7318 .6458 | .3677 .4134 .4517 .4835 .5098 | 4960 4180 3492 2893 2378 | \$253 7337 \$429 \$563 \$757 |
| 10 11 12 13 14 | 23885 25195 26427 27600 28727 | 1.3554 1.2679 1.1998 1.1476 1.1476 1.1079 | .9858 .7712 .5957 .4547 .3431 | .6378 .6836 .6846 .7016 .7151 | .2855 .2331 .1885 .1511 .1201 | .5646 .4844 .4089 .3406 .2804 | .5313 .5488 .5629 .5749 .5832 | .1939 .1570 .1262 .1007 .0798 | .4026 .3374 .2801 .3306 .1983 |
| 1-5 1-6 1.7 1.8 1.9 | 8.9819 5.0885 31933 38967 3.3990 | 1,0781 1,0560 1,0398 1,0380 1,0195 | 2560 1890 1380 0998 0714 | .7358 .7342 .7407 .7458 .7497 | .0948 .0742 .0577 .0445 .0341 | .2283 .1842 .1473 .1168 .0919 | .5903 .5959 .6003 .6036 .6068 | 0620 0491 0382 0294 02294 | 1825 1826 0978 0774 0608 |
| 2-0 2-1 2-2 2-3 2-4 | 3.5 0 0 6 3.6 0 1 8 3.7 0 2 5 3.8 0 3 0 3.9 0 3 4 | 1.0135 1.0092 1.0062 1.0042 1.0088 | .0506 .0355 .0246 .0169 .0115 | .7527 .7550 .7567 .7580 .7589 | .0260 .0196 .0147 .0110 .0082 | .0717 .0556 .0427 .0326 .0247 | .6082 .6097 .6108 .6116 .6183 | .0172 .0130 .0097 .0073 .0054 | 0474 0367 0282 0816 0163 |
| 2,5 2,6 2,7 2,8 2,9 | 4.0036 4.1038 4.2039 4.3039 4.3039 4.4040 | 10018 10012 10008 10005 10003 | 0077 0052 0034 0022 0014 | .7596 .7602 .7605 .7605 .7610 | .0060 .0044 .0032 .0023 .0023 | .0186 .0139 .0103 .0076 .0055 | .6127 .6131 .6133 .6135 .6136 | 0040 0029 0021 0015 0011 | 0193 0092 0068 0050 0037 |
| 30 31 38 33 34 | 4.5 0 4 0 4.5 0 4 0 4.7 0 4 0 4.8 0 4 0 4.9 0 4 0 | 10002 10001 10001 10001 10001 | .0009 .0006 .0004 .0008 .0008 | .7611 .7618 .7613 .7613 .7614 | .0019 .0008 .0006 .0004 .0003 | .0040 .0029 .0021 .0015 .0010 | .6137 .6138 .6138 .6139 .6139 | 0008 0005 0004 0003 0002 | 0027 0019 0014 0010 0007 |
| 3.5 3.6 3.7 3.8 3.9 | 5.0040 5.1040 5.2040 | 10000 10000 10000 | .0001 .0001 .0000 | .7614 .7614 .7614 .7614 .7614 .7614 | .0002 .0001 .0001 .0001 .0001 .0000 | .0007 .0005 .0005 .0002 .0002 | .6139 .6139 .6139 .6139 .6139 | 0001 0001 0001 0000 0000 | 0005 0003 0008 0008 0008 |
| 40 41 42 43 44 | | | | .7614 .7615 .7615 | .0000 0000 .0000 | .0001 .0001 .0000 | .6139 .6139 .6139 | 0000 0000 0000 | .0001 .0000 .0000 |

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TABLE I. - Concluded. LAMINAR BOUNDARY LAYER BEHIND SHOCK WAVE

(d) Solution of momentum and energy equation for $u_{\mu}/u_{\sigma} = 6.0$

| — — | | r | | | Prandtl numb | er, $\sigma = 0.72$ | | |
|---------------------------------|---|--|---|---|---|--|--|---|
| η | f | f' | ~£" | $\int_{\alpha}^{\alpha} r d\eta \qquad r$ | r -r ¹ | $\int_0^{\eta} d\eta$ | 8 •. | -5' |
| 0_0 .1 .2 .3 .4 | 00000 5597 10410 14499 17947 | 60000 51977 44383 37534 31603 | 81009 78721 72620 64078 54454 | 0.0000 .0913 .1791 .3602 .3388 .6 | 195 009 486 710 787 9619 | 00000 .0934 .1738 .2420 .8991 | 1.0000 .8683 .7417 .6242 .5183 | 1.3262 1.2991 1.2258 1.1208 .9963 |
| .5 .6 .7 .8 .9 | 2.0852 23307 25402 27214 28809 | 25643 22613 19418 16938 1.5048 | 44831 35937 28161 21642 16353 | .3 9 5 8 .5 .4 4 9 2 .4 .4 93 5 .4 .5 29 6 .3 .5 5 8 6 .2 | 816 .9681 871 .9118 004 .8187 239 .7094 586 .5983 | .3461 .3845 .4155 .4403 .4600 | 4252 3450 2771 3206 1742 | .8662 .7387 .6197 .5127 .4190 |
| 1.0 1.1 1.2 1.3 1.4 | 3.0 24 0 3.1 54 8 3.2 7 6 6 3.3 9 1 8 3.5 0 8 4 | 13631 18583 11818 11266 10873 | 12171 .8935 .6478 .4641 .3288 | .5817 .2 .5998 .1 .6138 .1 .6247 .0 .6330 .0 | 040 4941 594 4012 234 3212 947 2542 721 1991 | .4755 .4875 .4969 .5040 .5095 | .1364 .1060 .0818 .0526 .0476 | .3388 .2712 .2151 .1692 .1320 |
| 1.5 1.6 1.7 1.8 1.9 | 3,6097 3,7146 3,8180 39202 4,0217 | 1.0597 1.0403 1.0370 1.0179 1.0118 | 1 2304 1597 1096 .0744 .0500 | .6393 .0 .6440 .0 .6476 .0 .6502 .0 .6521 .0 | 545 .1545 409 .1189 305 .0908 226 .0687 166 .0517 | .5137 .5168 .5192 .5209 .5222 | .0360 .0270 .0201 .0149 .0109 | 1022 0785 0599 0453 0340 |
| 20 21 22 23 23 | 41 336 42232 43236 44 339 45 34 0 | 1.0077 1.0049 1.0031 1.0080 1.0012 | 00333 0219 0143 09092 0059 | .6536 .0 .6546 .0 .6554 .0 .6559 .0 .6559 .0 | 121 .0385 088 .0285 063 .0210 045 .0153 032 .0111 | .5231 .5238 .5243 '.5246 .5249 | .0080 .0058 .0042 .0042 .0021 | .0254 .0188 .0138 .0101 .0073 |
| 2.5 2.6 2.7 2.8 2.9 | 46241 47242 48242 49243 50243 | 1.0008 1.0005 1.0003 1.0003 1.0003 | .0037 .0023 .0015 .0009 .0005 | .6565 .0 .6567 .0 .6569 .0 .6570 .0 | 023 .0080 016 .0057 011 .0040 008 .0028 005 .0020 | .5251 .5252 .5253 .5253 .5253 .5254 | .0015 .0010 .0007 .0005 .0003 | .0053 .0038 .0027 .0019 .0013 |
| 30 31 38 33 34 | 51 243 52243 53243 53243 54243 55243 | 1.0001 1.0000 1.0000 1.0000 1.0000 | -0003 .0002 .0001 '.0001 '.0000 | .6571 .0 .6571 .0 .6571 .0 .6571 .0 .6571 .0 *.6571 .0 | 004 .0014 002 .0010 003 .0007 001 .0004 001 .0003 | .5254 .5254 .5254 .5255 .5255 | .0002 .0002 .0001 .0001 .0001 | .0009 .0006 .0004 .0003 .0002 |
| 35 36 37 38 39 | | | | .6572 .0 .6572 .0 .6572 .0 .6572 .0 .6572 .0 | 000 000 000 000 000 000 000 000 000 00 | .5255 .5255 .5255 .5255 .5255 .5255 | .0000 .0000 .0000 .0000 .0000 | .0001 .0001 .0001 .0000 .0000 |

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|-------------------|--------------------|-----------------------------|---------------------------------|--------------------------|-----------------------|
| | | 0.7 | 2 | 1. | 0 |
| $\frac{u_w}{u_e}$ | c _f √Re | $(\delta/\sqrt{\nu_e t})_i$ | <u>-√σc_fRe</u> Nu | $(\delta/\sqrt{v_et})_i$ | <u>-√σcf</u> Re Nu |
| 1.0 | 1.128 | 3.64 | 2.0 | 3.64 | 2.0 |
| 1.5 | 1.057 | | | 4.33 | 2.0 |
| 2.0 | 1.019 | 4.55 | 2.032 | 4.91 | 2.0 |
| 3.0 | .979 | | | 5.91 | 2.0 |
| 4.0 | .958 | 5.86 | 2.060 | 6.80 | 2.0 |
| 5.0 | .944 | | | 7.63 | 2.0 |
| 6.0 | .935 | 6.94 | 2.074 | 8.42 | 2.0 |

TABLE II. - SKIN-FRICTION AND HEAT-TRANSFER COEFFICIENTS

[Y = 1.4]

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Figure 1. - Coordinate systems used to study boundary layer behind shock advancing into stationary fluid.





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