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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

TECHNICAL NOTE 3325

SIMILAR SOLUTIONS FOR THE COMPRESSIBLE LAMINAR
BOUNDARY LAYER WITH HEAT TRANSFER

AND PRESSURE GRADIENT

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NACA

Washington February 1955

TN-3325

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SUMMARY

Stewartson's transformation is applied to the laminar compressible boundary-layer equations and the requirement of similarity is introduced, resulting in a set of ordinary nonlinear differential equations previously quoted by Stewartson, but unsolved. The requirements of the system are: Prandtl number of 1.0, linear viscosity-temperature relation across the boundary layer, an isothermal surface, and the particular distributions of free-stream velocity consistent with similar solutions. This system admits axial pressure gradients of arbitrary magnitude, heat flux normal to the surface, and arbitrary Mach numbers.

The system of differential equations is transformed to an integral system, with the velocity ratio as the independent variable. For this system, solutions are found for pressure gradients varying from that causing separation to the infinitely favorable gradient and for wall temperatures from absolute zero to twice the free-stream stagnation temperature. Some solutions for separated flows are also presented.

For favorable pressure gradients, the solutions are unique. For adverse pressure gradients, where the solutions are not unique, two solutions of the infinite family of possible solutions are identified as essentially viscid at the outer edge of the boundary layer and the remainder essentially inviscid. For the case of favorable pressure gradients with heated walls, the velocity within a portion of the boundary layer is shown to exceed the local external velocity. The variation of a Reynolds analogy parameter, which indicates the ratio of skin friction to heat transfer, is from zero to 7.4 for a surface of temperature twice the free-stream stagnation temperature, and from zero to 2.8 for a surface held at absolute zero where the value 2 applies to a flat plate.

INTRODUCTION

Factors that affect the development of laminar boundary layers are pressure gradient, Mach number, and heat transfer, plus the properties of the fluid under consideration. Since mathematical complexities preclude solutions of this problem in a completely general fashion, the literature consists largely of solutions treating particular combinations of these factors. For the flow of an ideal gas over a surface without pressure gradient, the remaining factors have been taken into account very completely by Crocco (ref. 2) and Chapman and Rubesin (ref. 3). For small pressure gradients, Low (ref. 4) has, by a perturbation analysis, treated the general problem of the isothermal surface. With the introduction of pressure gradients of arbitrary magnitude, other restrictions become necessary. The assumption of constant fluid properties (density, viscosity, etc.), for example, leads to the greatest simplification the separation of the momentum and energy equations. With this assumption. for a special case of a decelerating stream, Howarth (ref. 5) has obtained a series solution to the momentum equation. The introduction of a similarity concept (that the velocity or temperature profiles may always be expressed in terms of a single parameter) leads to a power-law free-stream velocity distribution. The momentum equation of this problem was first solved by Falkner and Skan (ref. 6), whose calculations were then improved by Hartree (ref. 7); the energy equation was later treated by Eckert (ref. 8) and others (refs. 9 and 10). For the same problem the restriction of constant fluid properties may be removed by alternatively requiring that the Mach number be essentially zero (ref. 11) or that the Mach number and the heat transfer be limited to small values (ref. 12).

Illingworth (ref. 13) and Stewartson (ref. 14) have demonstrated that, for an insulated surface in a fluid with a Prandtl number of 1.0, any compressible boundary-layer problem may be transformed to a corresponding problem in an incompressible fluid; the earlier solutions thus become applicable to certain compressible problems. For the case of heat flux across the surface, the transformation of Stewartson (ref. 14) with the concept of similarity introduced leads to a set of nonlinear ordinary differential equations previously quoted (ref. 14), but unsolved. Solutions to this set of equations, which are presented herein,

The principal developments of this paper, which is part of the Doctoral Dissertation of the senior author (ref. 1), were carried out under the stimulus and guidance of Professor Luigi Crocco and the sponsorship of the Daniel and Florence Guggenheim Foundation. The final analysis and the computations were completed at the NACA Lewis laboratory during the Spring of 1954.

are applicable to flows at arbitrary Mach number, pressure gradients of arbitrary magnitude (but of a form consistent with the requirements of similarity), and arbitrary but constant wall temperature.²

Since free-stream velocity distributions of the form required by similarity are not generally encountered in practice, the utility of these solutions is principally as follows: (1) the effects of pressure gradient, wall temperature, and Mach number may be viewed qualitatively; (2) the results may be used as a check on any approximate method (such as a Karman-Pohlhausen method) for reliability; (3) the flow to be solved may be divided intuitively into segments and the solution for each segment may be matched by some arbitrary technique; or (4) the results may be used to construct a new simple method (of the integral type) for the calculation of the laminar compressible boundary layer with heat transfer. This latter analysis has been carried out, utilizing the solutions herein given, and is presented in reference 1.

STEWARTSON'S EQUATIONS

Boundary-Layer Equations

The equations of the steady two-dimensional compressible laminar boundary layer for perfect fluids are:

Continuity:

$$\frac{\partial x}{\partial y} (\delta n) + \frac{\partial y}{\partial y} (\delta x) = 0$$
 (1)

Momentum:

$$\rho u \frac{\partial u}{\partial x} + \rho v \frac{\partial u}{\partial y} = -\frac{\partial x}{\partial x} + \frac{\partial y}{\partial y} \left(\mu \frac{\partial u}{\partial y} \right)$$
(2)

²Since this writing, further calculations, which are closely related to the present investigation, have been published by Levy (ref. 15). Solutions to the equations treated herein were obtained in that report. The present investigation includes ranges of variables not treated in ref. 15: for example, favorable pressure gradients applicable to supersonic nozzles and values of adverse pressure gradients including that causing separation. For adverse pressure gradients, the problems of uniqueness and multiple solutions are also considered in some detail. The solutions of ref. 15 were obtained by means of a differential analyzer, whereas the present solutions were obtained by digital calculation and are presented in tabular form.

Energy:

$$\rho u \frac{\partial h}{\partial x} + \rho v \frac{\partial h}{\partial y} = u \frac{\partial p}{\partial x} + \frac{\partial}{\partial y} \left(\frac{\mu}{Pr} \frac{\partial h}{\partial y} \right) + \mu \left(\frac{\partial u}{\partial y} \right)^2$$
 (3)

All symbols are defined in appendix A.

The viscosity law to be assumed is

$$\frac{\mu}{\mu_0} = \lambda \frac{t}{t_0} \tag{4}$$

Equation (4) is of the form taken by Chapman and Rubesin (ref. 3), except that the reference conditions (μ_0,t_0) are free-stream stagnation values, since in the presence of pressure gradient the local "external" values are not constant along the outer edge of the boundary layer. The constant λ is used to match the viscosity with the Sutherland value at a desired station. If this station is taken to be the surface, assumed to be at constant temperature, the result is

$$\lambda = \sqrt{\frac{t_w}{t_0}} \left(\frac{t_0 + k_{gu}}{t_w + k_{gu}} \right)$$
 (5)

where $k_{su} = Sutherland's constant$ (for air, $k_{su} = 216^{\circ}$ R). The viscosity law of equations (4) and (5) was demonstrated to be adequate for a flat plate (ref. 3) by comparison with the more exact calculations of reference 2. In the present case no such comparison is available.

Stewartson's Transformation

A slight modification of Stewartson's transformation may be written

$$dX = \lambda \frac{a_{\Theta}}{a_{O}} \frac{p_{\Theta}}{p_{O}} dx$$

$$dY = \frac{\rho}{\rho_{O}} \frac{a_{\Theta}}{a_{O}} dy$$

$$U \equiv \psi_{Y}$$

$$V \equiv -\psi_{X}$$
(6)

where the stream function is defined by

$$\psi_y = \frac{\rho u}{\rho_0}$$

$$\psi_{\mathbf{X}} = -\frac{\rho_{\mathbf{V}}}{\rho_{\mathbf{0}}}$$

The transformed quantities are now represented by upper-case letters (X,Y,U,V), and the subscript e refers to local conditions at the outer edge of the boundary layer (external). The subscript 0 refers to free-stream stagnation values. From the preceding transformation, a useful relation between the transformed and physical velocities is

$$U = \frac{\mathbf{a}_0}{\mathbf{a}_0} \mathbf{u}.$$

If equations (4) and (6) are applied to the boundary-layer equations (1), (2), and (3), and if Pr and c_p are taken to be constant (but it is not yet required that Pr = 1), there result

$$\mathbf{U}_{\mathbf{Y}} + \mathbf{V}_{\mathbf{Y}} = \mathbf{0} \tag{7}$$

$$UU_{X} + VU_{Y} = U_{\Theta}U_{\Theta_{X}}(1 + S) + \nu_{O}U_{YY}$$
 (8)

$$US_{X} + VS_{Y} = v_{0} \left\{ \frac{S_{YY}}{Pr} - \frac{1 - Pr}{Pr} \left(\frac{\frac{\gamma - 1}{2} M_{\Theta}^{2}}{1 + \frac{\gamma - 1}{2} M_{\Theta}^{2}} \right) \left[\left(\frac{U}{U_{\Theta}} \right)^{2} \right]_{YY} \right\}$$
(9)

where the enthalpy function S is defined for convenience as

$$S = \frac{h_s}{h_0} - 1 \tag{10}$$

and h_s is the local stagnation enthalpy.

The boundary conditions applicable to the system (7) to (9) are:

$$U(X,0) = 0$$

$$V(X,0) = 0$$

$$S(X,0) = S_{W} \text{ or } \left[\frac{\partial S}{\partial Y}(X,0) = \left(\frac{\partial S}{\partial Y}\right)_{W}\right]$$

$$\lim_{Y\to\infty} S = 0$$

$$\lim_{Y\to\infty} U = U_{\Theta}(X)$$

$$1 \text{ im } U = U_{\Theta}(X)$$

The solution S=0 and the resultant continuity and momentum equations (7) and (8) make up the extremely useful correlation developed by Stewartson between compressible and incompressible boundary layers on insulated surfaces with Pr=1. Another special case is that of $U_{e_X}=0$. Then, if Pr=1, the relation $S=S_w\left(1-\frac{U}{U_e}\right)$ satisfies equation (9); this is Crocco's integral of the energy equation for the flat plate (ref. 2).

Similarity Requirements

When a pressure gradient exists and the surface is not insulated, it is necessary to find a means of solving the system (7) to (9) subject to the boundary conditions (11). To this end, the question will be asked: Under what conditions can this system be reduced to a system of ordinary differential equations by the assumption that the boundary-layer profiles are functions of a similarity variable η and that the wall temperature is constant? This question may be resolved by inserting the following assumed relations into the system (7) to (9) and observing the conditions required for obtaining ordinary differential equations:

$$\Psi = AX^{a}U_{e}^{p} f(\eta)$$

$$Y = BX^{b}U_{e}^{q}\eta$$

$$S = S(\eta)$$
(12)

where A,B,a,b,p, and q are undetermined constants. This procedure has been carried out by Li and Nagamatsu (ref. 16) for Pr = 1. In that analysis it was concluded that four classes of similar solutions

are possible. It has been pointed out (ref. 17) that three of these four classes can be reduced identically to the case requiring that

$$U_{e} = CX^{m} \tag{13}$$

while the remaining case requires that

$$U_{\Theta} = C_{1} \exp \left[C_{2} X \right] \tag{14}$$

When equations (12) are used in the form

$$\psi = f(\eta) \sqrt{\frac{2\nu_0 U_{\Theta} X}{m+1}}$$

$$\eta = Y \sqrt{\frac{m+1}{2} \frac{U_{\Theta}}{\nu_0 X}}$$
(15)

the system of ordinary differential equations corresponding to the power-law velocity distribution of equation (13) may be written

$$f''' + ff'' = \beta(f'^2 - 1 - S)$$

$$S'' + PrfS' = (1 - Pr) \left[\frac{(\gamma - 1)M_{\Theta}^2}{1 + \frac{\gamma - 1}{2} M_{\Theta}^2} \right] (f'f''' + f''^2)$$
(16)

The pressure-gradient parameter β is defined as $\beta=\frac{2m}{m+1},$ and the velocity ratio is $U/U_e=u/u_e=f',$ where primes denote differentiation with respect to $\eta.$

The boundary conditions are:

$$f(0) = f'(0) = 0$$

$$S(0) = S_{W}$$

$$\lim_{\eta \to \infty} f' = 1$$

$$\lim_{\eta \to \infty} S = 0$$

$$(17)$$

Since M_{Θ} may, in general, be a function of x, the right member of the energy equation is not yet dimensionally consistent with the left member for arbitrary M_{Θ} and Pr.

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It was shown in reference 17 that, for the exponential case (eq. (14)) with $C_2 > 0$, the system (7) to (9) can be reduced to the ordinary differential equations (16), but with $\beta = 2$. For $C_2 < 0$, the f'" term in equations (16) is replaced by -f'". In this case, with S = 0, it can be shown that, because of the sign of the f'" term, no solution is possible in which the velocity ratio approaches its boundary condition smoothly. A question is thus raised as to the validity of any possible solution for $C_2 < 0$ regardless of the value of S. For the remainder of this paper this class will be omitted from consideration.

Corresponding analyses for incompressible flow, including conditions for similarity and the case of the exponential free-stream velocity, have been made by Mangler (ref. 18) and Goldstein (ref. 19), respectively. As previously mentioned, the right member of the energy equation (16) must be zero or a function of η to be consistent with the left member. This may be achieved in the following ways: (1) the external Mach number may be a constant other than zero, (2) the external Mach number may be zero, (3) the Prandtl number may equal 1, (4) the factor $\frac{(\gamma - 1)M_{\Theta}^2}{1 + \frac{\gamma - 1}{2}M_{\Theta}^2} \approx 2$ corresponding to hypersonic flow, or (5)

the ratio of specific heats γ may equal 1.

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The case of constant external Mach number is the flat-plate problem $(\beta=0)$ and, the solution to the momentum equation being known, the energy could be integrated directly. The flat-plate problem has already been solved with great accuracy and completeness by Crocco (ref. 2). If the pressure gradient is small enough, it may be reasonable to consider M_{Θ} constant in the energy equation in spite of the gradient, but to retain the pressure-gradient parameter in the momentum equation. However, this problem is treated more completely by the analysis of reference 4.

The case $M_{\Theta}=0$ (with arbitrary β) produces the equations of Levy and Seban (ref. 20). In that analysis approximate solutions were obtained by the assumption of simple forms for the velocity and temperature profiles which contained undetermined coefficients. These coefficients were then evaluated by use of the boundary conditions. Because the actual profiles cannot be simply represented, this method is not reliable in some ranges even if the Mach number is nearly zero. Brown and Donoughe (ref. 11) also considered the low Mach number problem with variable fluid properties and $Pr_{W}=0.7$. The system of equations encountered in that analysis is much more complicated than the present system because of the power-law viscosity, conductivity, and specificheat relations used. These refinements do not alter the effects of omitting the viscous-dissipation and compressive-work terms, which may be significant at higher Mach numbers.

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The case of hypersonic flow requires the introduction of the effects of displacement thickness upon pressure gradient, such as have been evaluated by Lees and Probstein (ref. 21), for example. This case will not be treated herein.

The possibility of assuming $\gamma=1$ does not simplify the equations beyond the assumption of Mach number zero. For most gases, the assumption of $\gamma=1$ is physically unreasonable. Therefore, this case does not appear to warrant further consideration.

If strong pressure gradients and reasonably high Mach numbers are to be considered, it thus appears desirable to restrict the similarity system to Pr = 1, with the result that

$$f''' + ff'' = \beta(f'^2 - 1 - S)$$
 (18a)

$$S'' + fS' = 0$$
 (18b)

with the boundary conditions (17). Equations (18) were derived by Stewartson by assigning similarity relations corresponding to (15) to the system (7) to (9) with Pr = 1; however, no solution was indicated.

The comparison between assuming that $M_{\rm e}=0$ (case (2)) or that ${\rm Pr}=1$ (case (3)) may perhaps be indicated by examination of the solutions to the insulated flat-plate problem, which include effects of both Prandtl number and Mach number (ref. 2). If $M_{\rm e}=0$, the viscous-dissipation and compressive-work terms are omitted in equation (3). Then the predicted temperature profile is a constant, rather than the correct variation from free-stream static to recovery temperature at the wall. However, if ${\rm Pr}=1$ is assumed, a constant stagnation temperature is predicted, rather than the actual slight variation in this quantity. The latter discrepancy is small compared with the former.

METHOD OF SOLUTION

Equations (18) with boundary conditions (17) comprise the system to be solved for the dependent variables $f(\eta)$ and $S(\eta)$. Because of the nonlinearity of the system, its high order (fifth), and its classification as a "two-point boundary-value problem," no standard integration methods will yield results expressible in closed form. Methods applicable to equations of this type may be classified as either (1) forward integrations or (2) integrations by methods of successive approximations.

By "forward integration" is meant the progressive integration of the equations from one (initial) boundary to the other. For this purpose several sets of initial values of the derivatives are assumed.

Then the final boundary values obtained are compared with those specified and, after interpolation of the initial values, this trial-anderror process is repeated until the final boundary conditions are satisfied. The integrations may be carried out by the use of either an analog computer (mechanical or electrical) giving continuous integrals or by digital computations involving finite-difference integration. Although generally applicable, a disadvantage associated with forward integration of nonlinear equations is the lack of any inherent convergence mechanism. Thus, the approach to the correct initial values depends almost entirely on the intuition and experience of the one performing the calculations. This method is particularly troublesome for a problem with more than one dependent variable since evidence for the fitness of a given initial value may be obscured by a poor selection of the corresponding initial value of another dependent variable. Furthermore, when an analog computer is employed the accuracy is generally limited, particularly for nonlinear equations where in certain regions the results tend to be highly sensitive to the chosen initial values. If digital computation is utilized to obtain a desired degree of accuracy, the procedure may become excessively tedious.

Successive approximation methods generally assume an entire function for the dependent variables (satisfying as many of the boundary conditions as possible) rather than only the initial derivatives. Then, by use of the differential equations, a procedure is developed for estimating the error as a function of the independent variable(s). This error is applied to the original choice and the process is repeated until satisfactory convergence occurs. An example of a method of successive approximation is Picard's method.

A difficulty shared by both these methods arises when the range of integration is infinite. Then it is necessary to decide upon a finite value of the independent variable at which the boundary conditions may be approximately satisfied and the degree to which they may be satisfied. This suggests the desirability of changing to an independent variable so that only a finite range of integration is required. In the present problem this change of variables can be achieved by following a method used by Crocco for the solution of the compressible flat-plate boundary layer (ref. 2). The concept is advanced that the velocity is a more suitable independent variable since it is bounded. This concept leads to a set of equations conveniently handled by a method of successive approximations.

Transformation to Velocity Plane

To accomplish the transformation to the velocity ratio f' as the independent variable, the following identity may be used:

$$\frac{d}{d\eta} \equiv f'' \frac{d}{df'} \tag{19}$$

This identity may be applied to f" and f as follows:

$$f''' = f'' \frac{df''}{df'}$$

$$f = \int_{0}^{\eta} f' d\eta = \int_{0}^{f'} \frac{f' df'}{f''} = \int_{0}^{f'} \frac{\xi d\xi}{f''(\xi)}$$
(20)

where the dummy variable of integration is ξ , and $f''(\xi)$ represents the functional relationship between f'' and f', that is, f''(f'). The primes continue to denote differentiation with respect to η .

Inserting equations (20) into the momentum equation (18a) results in

$$\frac{df''}{df'} = -\int_{0}^{f'} \frac{\xi d\xi}{f''(\xi)} + \beta \frac{f'^{2} - 1 - S}{f''}$$
 (21)

which satisfies the following condition at f' = 0 required by the momentum equation:

$$f_{W}^{""} = -\beta(1 + S_{W})$$
 (22)

Now, if equation (21) is integrated once with respect to f' and if the limits of integration are chosen so that $(f'')_{f'=1} = 0$, the result is

$$f'' = \int_{\mathbf{f}'}^{1} d\xi_1 \int_{0}^{\xi_1} \frac{\xi d\xi}{f''(\xi)} - \beta \int_{\mathbf{f}'}^{1} \frac{\xi^2 - 1 - S(\xi)}{f''(\xi)} d\xi$$
 (23)

By inverting the order of integration (or by integrating by parts) the double integral may be reduced to two single integrals, resulting in:

$$f_{j+1}''(f') = (1-f') \int_{0}^{f'} \frac{\xi d\xi}{f_{j}''(\xi)} + \int_{f'}^{1} \frac{(1-\xi)\xi d\xi}{f_{j}''(\xi)} - \beta \int_{f'}^{1} \frac{\xi^{2}-1-S_{j}(\xi)}{f_{j}''(\xi)} d\xi$$
(24)

Equation (24) is the form of the momentum equation as it will be used in this report. The subscript j is the iteration number in the method of successive approximations.

A corresponding form of the energy equation is obtained by writing equation (18b) as

$$\frac{S''}{S'} = -f$$

and integrating with respect to \,\ \eta, \to get

$$\ln S' = - \int f d\eta + constant$$
 (25)

Equation (18a) may be written

$$f d\eta = -\frac{f'''}{f'''} d\eta + \beta \frac{(f'^2 - 1 - S)}{f''} d\eta$$
$$= -\frac{df''}{f'''} + \beta \frac{(f'^2 - 1 - S)}{(f'')^2} df'$$

Substitution of this expression into equation (25) results in .

ln S' =
$$\int \frac{df''}{f''} - \beta \int \frac{\xi^2 - 1 - S(\xi)}{[f''(\xi)]^2} d\xi + constant$$

or the equivalent expression

$$S' = - C_3 f'' J(f')$$
 (26)

where

$$J(\xi) = \exp \left[-\beta \int_{0}^{\xi} \frac{\xi_{1}^{2} - 1 - S(\xi_{1})}{\left(f''(\xi_{1}) \right)^{2}} d\xi_{1} \right]$$

If this expression is integrated once again and the boundary conditions $S(0) = S_w$, $(S)_{f'=1} = 0$ are required, the result is

$$\frac{S_{j+1}}{S_w} = \frac{\int_{f'}^{1} J_j(\xi) d\xi}{\int_{0}^{1} J_j(\xi) d\xi}$$
(27)

Inspection of equations (24) and (27) indicates that the integrals to be evaluated are singular, or indeterminate, at the upper limit. To evaluate these integrals, closed-form expressions must be obtained for the integrands in this range. This requires knowledge of the solution of the system (18) for large η (near f'=1). This "asymptotic solution" and its development are given in appendix B. The results show that equation (24) can be used in its present form, but that equation (27) must be modified to

$$\frac{S_{j+1}}{S_{w}} = \frac{\varepsilon J_{j}(1-\varepsilon) + \int_{f'}^{1-\varepsilon} J_{j}(\xi)d\xi}{\varepsilon J_{j}(1-\varepsilon) + \int_{O'}^{1-\varepsilon} J_{j}(\xi)d\xi}$$
(28)

where ϵ is an arbitrary small quantity ($\epsilon << 1$). In this form the singularity has been removed. Equations (24) and (28) comprise the system used in the present investigation. The convergence of this system is discussed in appendix C, and the method of calculation in appendix D.

PROPERTIES OF SOLUTIONS

In the following sections the solutions obtained in this study are presented and their properties are discussed. The two parameters defining a case are $S_{\rm w}$ and β . The enthalpy function evaluated at the wall $S_{\rm w}$ determines the wall temperature through the relation

$$t_{\mathbf{W}} = t_0(1 + S_{\mathbf{W}}) \tag{29}$$

Thus, S_W = -1 corresponds to a wall temperature of absolute zero, and S_W = 1 corresponds to a wall at twice the free-stream stagnation temperature. The case S_W = 0 corresponds to a wall at the free-stream stagnation temperature, which for Pr = 1 is the case of an insulated surface.

The pressure-gradient parameter β is related to the exponent m of the velocity distribution in the transformed plane $U_\theta=CX^m$ through the relation

$$\beta = \frac{2m}{m+1}$$

For a velocity distribution of this form, m can be represented as

$$m = \left(\frac{u_{\Theta x}}{u_{\Theta}}\right) \frac{t_{O}}{t_{\Theta}} \left(a_{\Theta} p_{\Theta}\right)^{-1} \int_{O}^{x} a_{\Theta} p_{\Theta} dx \qquad (30)$$

It is apparent that $\beta<0\ (m<0)$ corresponds to an unfavorable gradient; $\beta=0\ (m=0)$ corresponds to flat-plate flow; and $\beta=2\ (m=\infty)$ corresponds to an infinitely favorable pressure gradient. Stewartson (ref. 14) has shown that $\beta=1\ (m=1)$ corresponds to flow in the immediate vicinity of a stagnation point for two-dimensional flow, as in the incompressible case. It can be shown that the case of a stagnation point in axisymmetric flow can be transformed to the solution for $\beta=1/2$ (ref. 22). An approximate method for relating β to more general physical flows is given in reference 1. Values of β of the order of magnitude ± 0.3 correspond to flows over supersonic wings, and a typical nozzle with an exit Mach number of about 2.5 might produce a value of β of about 1.5. In the present investigation, solutions are

found for pressure gradients ranging from that causing separation to the infinitely favorable gradient and for wall temperatures from absolute zero to twice free-stream stagnation temperature.³

All solutions are presented in tabular and graphic form. Table I shows the values of f, f', f", S, and S' tabulated against η . From these values and equations (18) the quantities f'" and S" can be easily calculated. Table II presents a summary of the values of $f_W^{"}$ (related to wall shear) and $S_W^{"}$ (related to heat transfer) from table I, as well as the Reynolds analogy parameter $C_f Re_W/Nu$, which represents the ratio of skin-friction to heat-transfer effects. Certain other quantities of interest cannot be tabulated in general, but can be easily calculated from the following formulas:

Static-temperature ratio:

$$\frac{t}{t_e} = \left(1 + \frac{\gamma - 1}{2} M_e^2\right) (1 + S) - \frac{\gamma - 1}{2} M_e^2 f^{-2}$$
 (31)

or, with the static temperature t referred to the free-stream stagnation temperature t_0 ,

$$\frac{\mathbf{t}}{\mathbf{t}_0} = (1 + S) - \left(\frac{\frac{\gamma - 1}{2} M_{\Theta}^2}{1 + \frac{\gamma - 1}{2} M_{\Theta}^2}\right) f^{1/2}$$
 (32)

Flux density:

$$\frac{\rho u}{\rho_{\Theta} u_{\Theta}} = \frac{f'}{\left(1 + \frac{\gamma - 1}{2} M_{\Theta}^2\right) (1 + S) - \frac{\gamma - 1}{2} M_{\Theta}^2 f'^2}$$
(33)

Uniqueness

For $\beta < 0$, $S_w = 0$, Hartree (ref. 7) first observed that the boundary conditions (17) are not sufficient to determine a unique solution.

 $^{^3}$ It should be noted that all but one of the presented solutions for $S_w = 0$ are those first obtained by Hartree (ref. 7) for the problem of Falkner and Skan (ref. 6). As a further check on the present method, the solutions for $\beta = 1.6$ and 2.0 with $S_w = 0$ were obtained independently in the present investigation; these values agree very well with those of Hartree.

Thus, there is not a unique value of $f_W^{"}$ for a given β . In studying the uniqueness, it is useful to consider the following expression for velocity ratio (for any S_W) valid for large η :

$$f' = 1 + \left[\alpha_1(\eta - x)^{-(2\beta+1)} + \frac{\alpha_3}{2}(\eta - x)^{-1}\right] \exp\left[-\frac{(\eta - x)^2}{2}\right] + \alpha_2(\eta - x)^{2\beta}$$
(34)

where α_1 , α_2 , α_3 , and x are integration constants (see appendix B). In case of $S_w = 0$, α_3 is also equal to zero; however, this does not change the uniqueness problem, which is independent of wall temperature. For $\beta > 0$, α_2 is necessarily zero in order to satisfy the boundary condition $\lim_{n \to \infty} f' = 1$. For continuity in β , Hartree then selected the $\eta \to \infty$

asymptotic solution with $\alpha_2 = 0$ for $\beta < 0$.

Another important result of the asymptotic solution is that the

integral
$$\int_0^\infty \left(1 - \frac{\rho u}{\rho_e u_e}\right) d\eta$$
, related to the displacement thickness,

can be shown to become infinite for $\alpha_2 \neq 0$. This result is contrary to the concept of a thin layer outside of which the viscous effects may be neglected. A further effect of the α_2 term on the solution can be observed by examination of the dimensionless quantity f'''/ff'' (suggested by Professors L. Crocco and L. Lees), in which f''' represents the net viscous forces acting on the fluid element and f'' is proportional to the velocity gradient (shearing flow). It can be shown that for $\alpha_2 = 0$

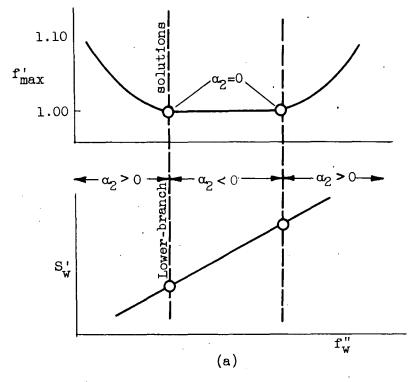
$$\lim_{\eta \to \infty} \left(-\frac{\tilde{\mathbf{f}}'''}{\tilde{\mathbf{f}}\tilde{\mathbf{f}}''} \right) = 1$$

while for $\alpha_2 \neq 0$

$$\lim_{\eta \to \infty} \left(-\frac{\mathbf{f}'''}{\mathbf{f}\mathbf{f}''} \right) = \lim_{\eta \to \infty} \frac{1 - 2\beta}{(\eta - \kappa)^2} = 0$$

Solutions with $\alpha_2=0$ retain the numerator and the denominator of the ratio $-\tilde{f}'''/\tilde{f}\tilde{f}''$ to the same order of magnitude, while if α_2 is different from zero a solution results wherein the magnitude of the net viscous forces in the asymptotic region is small compared with the magnitude of the shearing flow set up by their action. Thus, in order to retain both effects of viscosity to the same order of magnitude, α_2 must be taken equal to zero, as was done by Hartree. The solution thus obtained will be termed the "viscid" solution.

Another feature of solutions with α_2 different from zero is the analytical result that the velocity ratio in the outer portion of the boundary layer may exceed unity. For example, if α_2 is not zero, equation (34) shows that for large η the α_2 term of the velocity ratio expression is dominant, and thus (f'-1) is necessarily of the same sign as α_2 . That is, for positive α_2 , the velocity ratio approaches unity from above; this phenomenon will be termed "velocity overshoot." Since, for a given β and S_w in this range, each of these various solutions has associated with it a different set of values of f_w and S_w , one of these parameters, say f_w , can be conveniently used in place of α_2 to identify the various solutions. This infinite set of solutions can be represented as in sketch (a) for a typical (cold wall) case.



It is seen that there are a maximum and a minimum shear (represented by $f_W^{"}$) and heat transfer (represented by $S_W^{"}$) that can satisfy the equations without incurring velocity overshoot. These distinct solutions (circled points in sketch (a)) correspond to $\alpha_2 = 0$, 4 the viscid solutions; that

 $^{^4\}mathrm{In}$ the evaluation of the singularities of the integrals required , for the method of successive approximations, α_2 was taken to be zero. Hence, solutions for $\alpha_2 \neq 0$ were obtained by forward integration (appendix D), although the numerical values of α_2 were not determined.

with the lower shear is designated the "lower-branch" solution. The behavior of the calculated family of solutions is presented in figure 1 for S_{W} = -0.8 and β = -0.325, -0.3285, and -0.336. For a given value of S_{W} , as β is decreased the two viscid solutions approach each other. At a value of β to be designated β_{min} , these two solutions become identical and for $\beta < \beta_{min}$, no viscid solution exists. For negative β , only the viscid solutions will be considered in the remainder of this report.

With regard to the physical significance of the double solution, it may be noted that for adverse pressure gradients ($\beta < 0$) a real flow cannot completely reproduce the similar solution because $U_{\theta}(0) = \infty$ would be involved. However, a pressure field can, in principle, be applied to a developing boundary layer so that, after a phase of adjustment, the boundary layer would approach one of the similar solutions with $\beta < 0$ and stay quite close to it thereafter. It seems reasonable to believe that, depending on the way the pressure field is applied, one solution or the other corresponding to the same β could be approached after different adjustment phases. This result is exactly what Clauser (ref. 23) has found in his experimental work on similar turbulent boundary-layer flows.

Velocity and Temperature Profiles

The velocity and enthalpy-function profiles obtained from the tabulated solutions are presented as functions of η in figures 2 and 3, respectively. The distance y normal to the surface in the physical plane is related to the similarity variable η through equations (6) and (15), and may be expressed as

$$y = \frac{p_0 a_0}{p_{\theta} a_{\theta}} \sqrt{\frac{2}{m+1} \frac{v_0 X}{U_{\theta}}} \int_0^{\eta} \frac{t}{t_0} d\eta$$
 (35)

where t/t_0 is given by equation (32).

Velocity overshoot. - The velocity profiles shown in figure 2 indicate that for a given wall temperature the initial slope decreases as the pressure gradient becomes less favorable. For adverse pressure gradients an inflection point occurs within the boundary layer and moves outward as the gradient becomes more adverse. The velocity ratio varies monotonically from zero to the final value of 1.0 except for the cases of favorable pressure gradients with heated walls. Then the velocity ratio in the outer portion of the boundary layer reaches a maximum value greater than 1.0 before returning to its final value of 1.0 This type

of velocity overshoot was also obtained in the investigation of reference ll for favorable pressure gradients with heated walls and is to be distinguished from that associated with the nonunique inviscid solutions which occur only for adverse pressure gradients. When the wall is heated in a favorable pressure-gradient flow, the density within certain layers of the boundary layer is lowered so that, in spite of the viscous retardation, the flow is accelerated more than the external flow by the external pressure forces. Thus, a velocity greater than the external velocity may be obtained.

This phenomenon can be established by examination of equation (34) and the corresponding asymptotic expression for the enthalpy function (appendix B):

$$S = \alpha_3(\eta - x)^{-1} \exp \left[-\frac{(\eta - x)^2}{2} \right]$$
 (36)

For favorable pressure gradients, $\alpha_2 = 0$ as previously mentioned. Then, the α_3 term in equation (34) is dominant for large η . Thus, (f' - 1) and α_3 are of the same sign. Hence, for a heated wall (α_3 positive, eq. (36)), the velocity ratio must approach 1.0 from above.

Stagnation-temperature profiles. - Figure 3 shows that for Pr=1, the stagnation temperature varies monotonically across the boundary layer from the wall value to the free-stream value. For favorable pressure gradients with a cold wall, there is small variation with β of this distribution. The variation becomes more pronounced with an increase in wall temperature.

Boundary-layer thickness. - The velocity profiles (fig. 2) indicate that the boundary layer thickens as the wall shear stress diminishes. Also, for a given value of the pressure-gradient parameter β , the boundary layer, when considered in terms of η , thickens as the wall temperature is lowered. However, in the physical plane (in terms of y) because of the relation between y and η (eq. (35)) the trend is just the opposite. This emphasizes the necessity for careful consideration of the relation between the transformed quantities and their physical counterparts.

The thermal boundary layer also thickens as separation is approached. The relative thicknesses of the dynamic and thermal boundary layers may be conveniently observed from a plot of S against f' (fig. 4). Then if a fixed fraction of S_w , say 0.99, is chosen to define the thermal-layer thickness and if the same value of velocity ratio is taken to define the dynamic layer, it can be seen that, regardless of wall temperature, the thermal layer is thicker than the dynamic layer for favorable gradients and thinner for adverse gradients.

For Pr < 1 the relative magnitude of the dynamic thickness to the thermal thickness will be decreased, since the Prandtl number represents the ratio of viscous to thermal effects in the fluid.

Shear and Skin Friction

The shear distribution in the boundary layer is presented in figure 5, where f" is plotted as a function of η . The shear function f" is related to the shear stress τ through the expression

$$\tau \equiv \mu \frac{\partial u}{\partial y} = \left[\lambda \mu_0 U_e \left(\frac{t_e}{t_0} \right)^{\frac{2\gamma - 1}{\gamma - 1}} \sqrt{\frac{m + 1}{2} \frac{U_e}{v_0 X}} \right] f''$$
 (37)

For $\beta > 0$ the maximum shear is at the wall, whereas for $\beta < 0$ the point of maximum shear moves increasingly outward as the pressure gradient becomes more adverse.

The quantity that is of primary interest in boundary-layer calculations is the shear stress at the wall $\tau_{\rm w}$, which can be made dimensionless through the definition of a local skin-friction coefficient, producing the relation

$$C_{f} = \frac{\tau_{w}}{\frac{1}{2} \rho_{w} u_{e}^{2}} = f_{w}^{"} \left[2\lambda (1 + S_{w}) \right] \sqrt{\frac{m+1}{2} \frac{\nu_{0}}{U_{e}X}}$$
(38)

The factor (l + S_w) appears in equation (38) because of the use of ρ_w in the definition of C_f . Although this factor can be easily avoided, it is used later in evaluating a Reynolds number $Re_w = \frac{u_e x}{v_w}$ suitable for use in determining the heat transfer. An alternate form for

equation (38) is

$$\frac{C_{f}\sqrt{Re_{w}}}{2} = f_{w}^{"}\sqrt{\frac{m+1}{2}\frac{d \ln x}{d \ln x}}$$
 (38a)

It should be noted that, in equation (38a), fluid properties are evaluated at the wall temperature. If the skin-friction coefficient and the Reynolds number were to be based on local free-stream fluid

properties, rather than on wall values, a factor of $\sqrt{\frac{\mu_w}{\mu_e}} \frac{t_e}{t_w}$ would appear in the right member of equation (38a). When this factor is evaluated using Sutherland's viscosity law, it varies from $\left(\frac{t_w}{t_e}\right)^{1/4}$ to $\left(\frac{t_w}{t_e}\right)^{1/4}$, depending on the temperature involved.

The quantity f_W'' is presented as a function of β and S_W in figure 6. It can be seen that heating the surface increases the sensitivity of the wall shear to pressure gradient, while cooling the wall has the opposite effect. A suggested physical interpretation for this trend is related to the effect of wall temperature on the mean density of the fluid within the boundary layer. For the heated wall, the boundary-layer density is less than the free-stream density, rendering the boundary-layer fluid more susceptible to free-stream acceleration forces than for the cold wall. Figure 6 shows further that a linear extension of the slope of the curve, f_W'' against β from $\beta=0$ to large positive β would grossly overemphasize the effects of favorable pressure gradient; while the same linear extension toward negative β would underemphasize the effects of adverse gradient.

In figure 6(b), the two viscid solutions, which occur for adverse pressure gradients for a given β and S_w , are plotted. It is seen that two solutions are given for even the insulated surface $(S_w \neq 0)$, although Hartree reported only one. In this case the lower-branch solution corresponds to negative wall shear stress (separated flow), which was not considered in reference 7. For heated walls $(S_w > 0)$ both solutions may be separated near β_{min} , while for cooled walls both solutions may be unseparated in this region. The physical interpretation of these double solutions has been discussed in the section UNIQUENESS.

Heat Transfer

The variation of heat transfer across the boundary layer is plotted in figure 7 in terms of the derivative of the enthalpy function $S' = \frac{dS}{d\eta}.$ This quantity is related to the stagnation enthalpy derivative in the physical plane by the expression

$$\frac{\partial}{\partial y} \left(\frac{h_{\mathbf{g}}}{h_{\mathbf{O}}} \right) = \left(\frac{\rho a_{\mathbf{g}}}{\rho_{\mathbf{O}} a_{\mathbf{O}}} \sqrt{\frac{m+1}{2} \frac{U_{\mathbf{g}}}{\nu_{\mathbf{O}} X}} \right) S' \tag{39}$$

These curves again indicate the thickening of the thermal layer as separation is approached. Furthermore, as separation is neared, the zone adjacent to the surface where S' is essentially constant spreads rapidly. This is a zone where the heat transfer is primarily by conduction because of the near zero velocities in the neighborhood of the surface.

The values of S' at the surface (S_W^+) are shown plotted as a function of pressure-gradient parameter β in figure 8 for constant wall temperatures. Two facts are noteworthy: (1) In the region of favorable pressure gradient, S_W^+ is nearly constant; (2) the heat transfer varies sharply near separation. From these facts the additional conclusion may be drawn that, if a linear extension of these curves is made with the slope at $\beta=0$, the result will seriously overemphasize the effects of a favorable pressure gradient or heat transfer and underestimate the effects for adverse pressure gradients. A similar influence of pressure gradient on skin friction has already been noted. A comparison of figures 6 and 8 indicates that the effect of pressure gradient on heat transfer is smaller than the corresponding effect upon wall shear.

As with the skin friction, it is convenient to define a dimensionless number from which the heat transfer may be determined. The Nusselt number is

$$Nu = \frac{x \left(\frac{\partial t}{\partial y}\right)_{w}}{t_{0} - t_{w}} = \left(-\frac{S_{w}'}{S_{w}}\right) \sqrt{Re_{w}} \sqrt{\frac{m+1}{2} \frac{d \ln X}{d \ln x}}$$

$$(40)$$

The quantity $(-S_W^*/S_W^*)$ is plotted in figure 9 for constant wall temperatures as a function of the pressure-gradient parameter β . The Reynolds number Re $_W$ is again defined in terms of wall properties.

Reynolds analogy. - From expressions (38a) and (40), a simple modified Reynolds analogy parameter is evaluated by

$$\frac{C_{f}Re_{w}}{Nu} = \frac{2f_{w}^{"}}{\left(-\frac{S_{w}^{'}}{S_{w}}\right)}$$
(41)

This quantity is the reciprocal of the usual Reynolds analogy quantity in order to avoid infinite values as separation is approached. It is plotted in figure 10 as a function of the pressure-gradient parameter β . These curves resemble the $f_W^{"}$ curves (fig. 6) because of the relatively small variation in magnitude of $S_W^{"}/S_W$ compared with that of

 f_W^* . The variation of $C_f Re_W/Nu$ is from zero to 7.4 for a surface of temperature twice the free-stream stagnation value and from zero to 2.8 for a surface held at a temperature of absolute zero, as shown in figure 10. This indicates the inadequacy of utilizing the flat-plate value of 2.0, as has often been done for estimates of heat transfer. Figure 10 is of particular use in evaluating the heat transfer for a problem when used in conjunction with simple methods for determining C_f , as proposed, for example, in reference 1.

SUMMARY OF RESULTS

From an analysis of the laminar compressible boundary layer based on Stewartson's transformation and including effects of heat transfer and pressure gradient, the following results were obtained:

- 1. If the condition of similarity is required and the Prandtl number is constant but different from 1.0, the external Mach number must be either zero, constant, or very large. If the Prandtl number is taken as 1.0, the Mach number may be arbitrary. The free-stream velocity distributions consistent with the similarity concept are either power-law or exponential distributions in the transformed coordinates. Since the exponential distribution appears to be limited to favorable gradients and in this range the problem may be reduced to a special case of the power-law distribution, the calculations have been based on the latter class.
- 2. For flows with favorable pressure gradients, unique solutions were obtained. For flows with adverse pressure gradients, two types of solution were obtained which have been identified as either essentially viscid or inviscid in the outer portions of the boundary layer. The inviscid solution sometimes involved velocity overshoot within the boundary layer. For favorable pressure gradients, the viscid solution is required by the boundary conditions. For adverse pressure gradients there are two viscid solutions; these correspond to the maximum and minimum wall shear, which exclude velocity overshoot.
- 3. For heated surfaces with favorable pressure gradients a velocity overshoot, which increases with increasingly favorable gradient, results within the boundary layer. This excess velocity is associated with the acceleration of a layer of fluid in the outer portion of the boundary layer, with density less than the external density. Since this layer is subject to the external pressure field and is restrained only slightly by the viscous forces acting on it, it is accelerated more than the external flow.
- 4. For a Prandtl number of 1.0, when the thicknesses of the dynamic and thermal boundary layers are defined by a fixed fraction

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(say 0.99) of the velocity ratio or stagnation-temperature-difference ratio, the thermal boundary layer is thicker than the dynamic layer for favorable pressure gradients and thinner for adverse gradients.

5. The variation of a Reynolds' analogy parameter is from zero to 7.4 for a surface of temperature twice the free-stream stagnation value and from zero to 2.8 for a surface held at a temperature of absolute zero, with the value 2.0 for the flat plate.

Lewis Flight Propulsion Laboratory
National Advisory Committee for Aeronautics
Cleveland, Ohio, October 15, 1954

APPENDIX A

SYMBOLS

The following symbols are used in this report:

sonic velocity C,C_1,C_2 , etc. arbitrary constants local skin-friction coefficient, $C_f = \frac{2\tau_w}{c_{o,u}^2}$ $C_{\mathbf{f}}$ specific heat at constant pressure c_p function related to stream function by $f = \psi \sqrt{\frac{m+1}{2\nu_0 U_e X}}$ f asymptotic function, $g = \tilde{f}_2^*$ g h enthalpy k thermal conductivity Sutherland's constant ksu local external Mach number, $M_{\Theta} = \frac{u_{\Theta}}{a_{\Omega}}$ M_{e} exponent from $U_{\Theta} = CX^{m}$ Nusselt number, Nu = $\frac{x(\frac{\partial t}{\partial y})_W}{t_0 - t_-}$ Nu Prandtl number, $Pr = \frac{\mu c_p}{k}$ Pr static pressure p Reynolds number, $Re_W = \frac{\rho_W u_e x}{\mu_-}$ Ře_w enthalpy function, $S = \frac{h_8}{h_0} - 1$ S t static temperature U transformed longitudinal velocity component,

 $U = \frac{ua_0}{a_0} = \psi_Y$

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u longitudinal velocity component

V transformed normal velocity component, $V = -\psi_X$

v normal velocity component

X transformed longitudinal coordinate, $X = \int_{0}^{x} \lambda \frac{p_{e}}{p_{0}} \frac{a_{e}}{a_{0}} dx$

x longitudinal coordinate

Y transformed normal coordinate, $Y = \int_0^y \frac{\rho a_0}{\rho_0 a_0} dy$

y normal coordinate

 α_1, α_2 , etc. integration constants in asymptotic solution

 β pressure gradient parameter, $\beta = \frac{2m}{m+1}$

 β_{min} minimum value of β corresponding to a viscid solution for a given wall temperature

 γ ratio of specific heats

ε arbitrary small quantity

η similarity variable, $η = \frac{Y}{X} \sqrt{\frac{m+1}{2} \frac{U_{\Theta}X}{v_{O}}}$

x integration constant in $f_{\eta} = \eta - x$

 $\lambda = \frac{(\mu/\mu_0)}{(t/t_0)} = \left(\frac{t_0 + k_{su}}{t_w + k_{su}}\right) \sqrt{\frac{t_w}{t_0}}$

μ dynamic viscosity

 ν kinematic viscosity, $\nu = \mu/\rho$

ρ mass density

 τ shear stress, $\tau = \mu \frac{\partial u}{\partial y}$

 ψ stream function: $\psi_{Y} = U$, $\psi_{X} = -V$

 Ω oscillation coefficient, eq. (C2)

ω damping coefficient, eq. (C3)

Subscripts:

e local flow outside boundary layer (external)

j result of jth iteration.

s stagnation value

w wall or surface value

O free-stream stagnation value

Other notations:

asymptotic quantity

primes denote differentiation with respect to η

APPENDIX B

ASYMPTOTIC SOLUTION

To evaluate the integrals in equations (24) and (27), it is necessary to have closed-form expressions for the integrands concerned, in the range of large η . This requires a solution of the system

$$f''' + ff'' = \beta(f'^2 - 1 - S)$$
 (18a)

$$S'' + fS' = 0 \tag{18b}$$

for large n, which is the asymptotic solution.

The asymptotic solution for f (designated \tilde{f}) is assumed to consist of a sum of terms, each smaller than the preceding. Only the first two terms will be discussed herein. The corresponding solution for the enthalpy term \tilde{S} is also obtained.

Let

$$\tilde{\mathbf{f}} = \tilde{\mathbf{f}}_1 + \tilde{\mathbf{f}}_2 \tag{B1}$$

where

$$\tilde{f}_2 \ll \tilde{f}_1$$

$$\tilde{f}_2^{\prime} \ll \tilde{f}_1^{\prime}$$

Now, since $\lim_{\eta \to \infty} (f') = 1$, let

$$\hat{\mathbf{f}}_1 = \eta - \mathbf{x} \tag{B2}$$

where \mathbf{x} is an undetermined constant. If \mathbf{f}_1 is inserted into (18), the corresponding enthalpy term \mathbf{S}_1 must be identically zero. Inserting equations (B1) and (B2) into equations (18) and dropping higher-order terms result in

$$\widetilde{\mathbf{f}}_{2}^{""} + (\eta - \varkappa)\widetilde{\mathbf{f}}_{2}^{"} = \beta \left[2\widetilde{\mathbf{f}}_{2}^{"} - \widetilde{\mathbf{S}}_{2}^{"} \right] \\
\widetilde{\mathbf{S}}_{2}^{"} + (\eta - \varkappa)\widetilde{\mathbf{S}}_{2}^{"} = 0$$
(B3)

The energy equation can be integrated directly to give

$$\tilde{S}_{2}' = Ce^{-\frac{(\eta - x)^{2}}{2}}$$

which integrates once again to the complementary error function (denoted cerf)

$$\tilde{S}_2 = -C \int_{\eta}^{\infty} e^{-\frac{(\eta - \kappa)^2}{2}} d\eta$$

or

$$\tilde{S}_{2} = \alpha_{3} \sqrt{\frac{\pi}{2}} \operatorname{cerf}\left(\frac{\eta - \kappa}{\sqrt{2}}\right)$$
 (B4)

If equation (B4) is now substituted into the momentum equation of equations (B3), with the notation

$$g(\eta) \equiv \tilde{f}_2$$

there results

$$g'' + (\eta - x)g' - 2\beta g = -\alpha_3 \sqrt{\frac{\pi}{2}} \beta \operatorname{cerf}\left(\frac{\eta - x}{\sqrt{2}}\right)$$
 (B5)

A particular integral to equation (B5) is

$$g = \frac{\alpha_3}{2} \sqrt{\frac{\pi}{2}} \operatorname{cerf}\left(\frac{\eta - \kappa}{\sqrt{2}}\right)$$
 (B6)

The complementary function can be found by noting that the homogeneous part of equation (B5) is Weber's equation. Hartree (ref. 7) gives the general solution for large values of the argument $(\eta - \varkappa)$ which can be written

$$g = \alpha_1(\eta - \kappa)^{-(2\beta+1)} \exp \left[-\frac{(\eta - \kappa)^2}{2} \right] + \alpha_2(\eta - \kappa)^{2\beta}$$
 (B7)

where α_1 and α_2 are undetermined constants.

For $\beta \geqslant 0$ it is clearly necessary to take $\alpha_2 = 0$ if the boundary condition $\lim_{\eta \to \infty} g = 0$ is to be applied. For $\beta < 0$ the

boundary condition does not require $\alpha_2 = 0$; this introduces a lack of uniqueness in this range. The significance of $\alpha_2 = 0$ was more fully discussed in the section UNIQUENESS.

Using the first term of the expansion for the complementary error function

$$\operatorname{cerf}\left(\frac{\eta-x}{\sqrt{2}}\right) = \left\{\sqrt{\frac{2}{\pi}} \left(\eta-x\right)^{-1} \exp\left[-\frac{(\eta-x)^2}{2}\right]\right\} \left[1-\frac{1}{(\eta-x)^2}+\cdots\right]$$
(B8)

and combining the preceding equations result in the following expressions:

$$\tilde{\mathbf{f}}' = 1 + \left[\alpha_{1}(\eta - \varkappa)^{-(2\beta+1)} + \frac{\alpha_{3}}{2}(\eta - \varkappa)^{-1}\right] \exp\left[-\frac{(\eta - \varkappa)^{2}}{2}\right] + \alpha_{2}(\eta - \varkappa)^{2\beta}$$
(34)

and

$$\tilde{S} = \alpha_3 (\eta - \kappa)^{-1} \left[\exp - \frac{(\eta - \kappa)^2}{2} \right]$$
 (36)

APPENDIX C

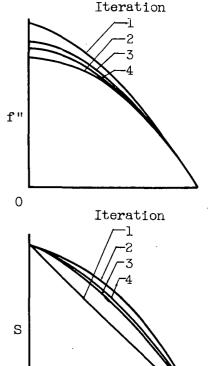
CONVERGENCE AND EXTRAPOLATION

The method of successive approximations used in solving equations (24) and (28) is as follows: Two functions $f_j''(f')$ and $S_j(f')$ are assumed and inserted into the right sides of equations (24) and (28). This produces two new functions, $f_{j+1}''(f')$ and $S_{j+1}(f')$ on the left.

The question of convergence is the first to consider. In reference 2, Crocco treated a momentum equation which was essentially equation (24) with $\beta=0.$ There it was shown that the result might converge to a pair of functions between which it would oscillate and of which the geometric mean was the proper solution. In practice, the use of the arithmetic mean was demonstrated to be adequate. In the same way in the present case, the property of oscillation cannot be developed analytically; however, it has been found

by trial that, if $\frac{f_{j+1}''+f_{j+1}''}{2}$ is used in place of f_{j+1}'' to obtain f_{j+2}'' , the oscillation is reduced and a convergence takes place. A typical result is shown in sketch (b).

When the value for \$\beta\$ for which a solution was sought was sufficiently positive, the enthalpy function \$S\$ also showed a tendency to oscillate. In these cases, applying the same averaging procedure to \$S\$ again improved the convergence. It was also \$O\$ found that convergence was improved if, in the intial assumed function for \$f''(f')\$, the



f'

(b)

slope $\left(\frac{df''}{df'}\right)_{W}$ was taken so that it satisfied equation (18a); that is,

$$\left(\frac{\mathrm{d}f''}{\mathrm{d}f'}\right)_{W} = \frac{-\beta(1+S_{W})}{f''_{W}}$$

When an iterative method is used to determine a function, it is always desirable to develop a method of extrapolating the result to correspond to a larger number of iterations than have actually been carried out. This cannot be done in an exact fashion unless a definite law of convergence is established. Recently, an extrapolation method was

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devised (ref. 24) which required four successive iterants for an arbitrary iterative computing scheme. The development assumed that the remaining error after any iteration consisted essentially of two terms, both of which damped by a factor ω with each iteration. The sign of one of these terms was assumed to change with each iteration. This method extrapolated a function by breaking it into n-l parts and treating it somewhat like an n-dimensional vector. The method has been demonstrated for Laplace's equation for which it was quite adequate. For nonlinear equations, however, the method is not as suitable.

In reference 1, a method requiring five successive iterants was developed which combined the method of reference 24 and the geometric mean rule. The function to be extrapolated is considered to be made up of a set of numbers F_i , where the subscript i identifies the particular component of the set. Then, the resulting relations for the i^{th} component of the extrapolated function F in terms of the preceding five iterants, $(F_i)_{j+4}$, where j is the iteration number, are:

$$F_{i} = \frac{1}{\Omega_{i}} \left[\frac{(F_{i})_{j+4} - \omega^{2}(F_{i})_{j+2}}{1 - \omega^{2}} \right]$$
 (C1)

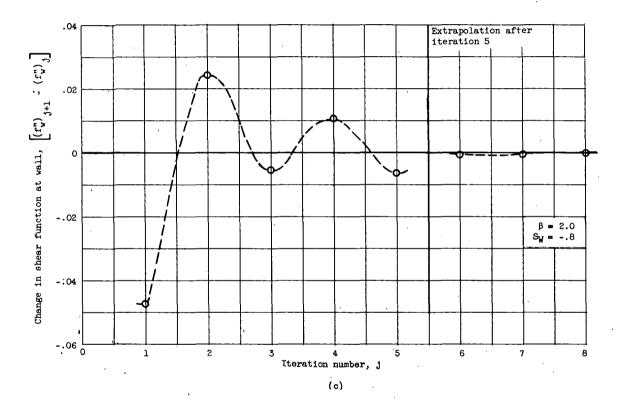
where the oscillation coefficient Ω_{i} is given by

$$\Omega_{1} = \sqrt{\frac{(F_{1})_{j+4} - \omega^{2}(F_{1})_{j+2}}{(F_{1})_{j+3} - \omega^{2}(F_{1})_{j+1}}}$$
(C2)

and the damping coefficient ω is

$$\omega^{2} = \frac{\sum_{i=1}^{n} \left[(F_{i})_{j+4} - (F_{i})_{j+2} \right] \left[\frac{(F_{i})_{j+2} - (F_{i})_{j}}{\left| (F_{i})_{j+2} - (F_{i})_{j} \right|} \right]}{\sum_{i=1}^{n} \left| (F_{i})_{j+2} - (F_{i})_{j} \right|}$$
(C3)

Application of this system was extremely effective. It generally reduced the oscillation remaining after five iterations by a factor of 10. A typical plot of the oscillation of f_W^* is indicated in sketch (c).



APPENDIX D

CALCULATION PROCEDURE

The successive approximation calculations were carried out by means of IBM Type 604 Calculating Punch machines. The program was coded for fixed-point calculation, with the standard Function-Generating control panel used, plus a control panel especially wired for rapid integration of quotients by a trapezoidal rule. The step size (in f') varied from a maximum value of 0.050 or 0.025 at f' = 0 to 0.00001 at f' = 0.9999, the total number of intervals being 122 in the former case and 236 in the latter. By doubling and halving the step size for a critical case, the results are judged to contain a maximum error of 0.0002. Comparison with solutions obtained by forward integration, for the same case, confirms this accuracy. A given iteration (utilizing the 0.050 step size) could be carried out in approximately $1\frac{1}{2}$ hours by an experienced machine operator. averaging and extrapolation techniques described in appendix C are used. 10 iterations generally would suffice for the accuracy desired. contrast with forward integration, this number of iterations is not a function of the experience of the person carrying out the calculations.

In the derivation of the integral relations (eqs. (24) and (27)), it was assumed that the velocity ratio varied smoothly and monotonically from zero at the wall to 1.0 at infinity. However, in the range $\beta>0$ and $S_{\rm w}>0$ (favorable pressure gradient and hot wall), the solution involves an increasing velocity ratio to a value greater than 1.0, followed by a smooth decrease to 1.0. Under these unusual circumstances, the method of successive approximation derived herein must be considerably modified if it is to be used at all. For these cases, forward integrations were performed by Dr. Lynn U. Albers.

Equations (18), together with the boundary conditions (17), constitute a nonlinear two-point boundary-value problem. Cases of this boundary-value problem were solved by forward integration, with the IBM Card-Programmed Electronic Calculator (CPC) used to integrate with five-point integration formulas.

For the cases where the solutions are not unique ($\beta < 0$), the solutions were obtained in two patterns: In one pattern, β and S_W were fixed and, for a set of values of $f_W^{"}$, the quantity $S_W^{"}$ was altered until boundary conditions at infinity were apparently satisfied. In the other pattern, $f_W^{"}$ and S_W were fixed and, for a set of values of negative β , the quantity $S_W^{"}$ was altered until boundary conditions at infinity were apparently satisfied. An attempt was made with both patterns to include the solution with the minimum value

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of the maximum velocity ratio f_{max} within the boundary layer. Except for those cases where no solution existed without velocity overshoot, this minimum value was 1.0.

The details of the integration method used are described very completely by Lynn U. Albers in an appendix to reference 25. The possible error contained in the results is indicated in the footnote to table I. Each trial run of a case required approximately 30 minutes. A person considerably experienced with the method of obtaining solutions by forward integration generally achieved convergence within 12 trials; however, tests indicate that this number is insufficient by a factor of the order of 2 if the person lacks experience.

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TABLE 1. - SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS. 1

	·····	β = -0.32	26, S _w = -1.	ō .				$\beta = -0.3657$	$S_{W} = -1.0$, , , , , , , , , , , , , , , , , , ,
$\overline{\eta}$	f	f'i	f"	S	S'	η	f	f'	f"	S	S'
0 .2 .4 .6 .8	.0000 .0001 .0004 .0014	.0001 .0009 .0029	0 .0016 .0065 .0145 .0258	-1.0000 9505 9009 8514 8019	0.2477 .2477 .2477 .2476 .2476	0 .2 .4 .6 .8	0 .0010 .0041 .0096 .0178	0 .0101 .0211 .0339	0.0 5 0 0 0 5 2 2 .0 5 8 6 .0 6 9 3 .0 8 4 1	-1.0000 9408 8817 8226 7636	0.2 9 5 8 .2 9 5 8 .2 9 5 6 .2 9 5 2 .2 9 4 4
1.0 1.2 1.4 1.6 1.8	.0034 .0070 .0129 .0220 .0352	.0135 .0232 .0369 .0550 .0781	.0403 .0580 .0788 .1026	7524 7029 6535 6042 5552	.2 4 7 5 .2 4 7 2 .2 4 6 8 .2 4 5 9 .2 4 4 5	1.0 1.2 1.4 1.6 1.8	.0295 .0452 .0660 .0928 .1268	.0678 .0905 .1180 .1510	.1028 .1252 .1509 .179.1 .2091	7049 6465 5886 5314 4754	.2931 .2909 .2877 .2832 .2771
2.0 2.2 2.4 2.6 2.8	.0536 .0783 .1105 .1516 .2026	.1067 .1413 .1821 .2291 .2822	.1578 .1882 .2196 .2506 .2799	5064 4582 4108 3645 3195	.2 4 2 4 .2 3 9 2 .2 3 4 3 .2 2 8 8 .2 2 0 8	2.0 2.2 2.4 2.6 2.8	.1691 .2211 .2837 .3582 .4454	.2347 .2856 .3421 .4035 .4686	.2395 .2690 .2956 .3174 .3324	4208 3679 3174 2697 2252	2690 2588 2461 2308 2131
3.0 3.2 3.4 3.6 3.8 4.0	23267 4937 43277 45277 4577	.3 4 0 9 2 .4 0 0 0 8 .4 7 0 0 8 .5 3 9 7 5 .6 7 3 7	.3056 .3260 .33692 .3434 .3377 .3220	2763 2353 1970 1619 1303 1025	.2108 .1985 .1839 .1672 .1488 .1292	3.0 3.2 3.4 3.6 3.8	.5 4 5 8 .6 5 9 8 .7 8 7 1 .9 2 7 3 1.0 7 9 3	.5359 .6035 .6694 .7315 .7881	.3388 .3354 .3216 .2983 .2668	1846 1482 1162 0889 0663	.1930 .1711 .1481 .1248 .1022
4.2 4.4 4.6 4.8 5.0	.9116 1.0645 1.2279 1.4004 1.5803	.7357 .7919 .8410 .8824 .9158	.2969 .2642 .2265 .1867 .1478	0786 0588 0427 0301 0206	.1 0 9 2 .0 8 9 7 .0 7 1 3 .0 5 4 8 .0 4 0 7	4.0 4.2 4.4 4.6 4.8	1,2421 1,4140 1,5935 1,7791 1,9693	.8379 .8799 .9140 .9406 .9605	.2299 .1904 .1513 .1154 .0843	0480 0337 0229 0151 0096	.0810 .0621 .0460 .0328 .0225
5.2 5.4 5.6 5.8 6.0	1.7662 1.9566 2.1502 2.3462 2.5437	.9417 .9610 .9748 .9843 .9905	.1 1 2 3 .0 8 1 8 .0 5 7 1 .0 3 8 3 .0 2 4 5	0137 0088 0055 0033 0020	.0 2 9 1 .0 2 0 1 .0 1 3 3 .0 0 8 5	5.0 5.2 5.4 5.6 5.8	2.1629 2.3589 2.5565 2.7551 2.9543	.9747 .9845 .9909 .9949 .9973	.0589 .0395 .0253 .0155 .0092	0059 0035 0020 0011 0006	.0149 .0095 .0058 .0034 .0019
6.2 6.4 6.6 6.8	2.7422 2.9414 3.1409 3.3405	.9944 .9967 .9980 .9988	.0152 .0088 .0050 .0028	0012 0007 0005 0004	.0031	6.0 6.2 6.4	3.1 5 3 9 3.3 5 3 8 3.5 5 3 7	.9987 .9995 .9999	.0052 .0028 .0014	0003 0001 0001	.0011 .0006 .0003

In accuracy of solutions obtained by the method of successive approximations is believed to be ± 0.0002 . Solutions by forward integration were obtained in two patterns (appendix D). Where β and S_W were initially fixed, the eigenvalues are believed to be correct to ± 0.0002 . Where f_W^{\parallel} and S_W were initially fixed, β and S_W^{\parallel} are believed to be correct to ± 0.0002 (except in the case of β = 0.2460, S_W = -0.4, where β and S_W^{\parallel} are believed to be correct to ± 0.002). The values in the tables are of comparable accuracy except at large η , where the entries may contain errors as large as twice the above amounts.

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.3884, S_{W} = -1.0$						
η	f	f'	f"	S	S¹		
0 .2 .4 .6 .8	0 .0028 .0113 .0259 .0470	0 .0282 .0574 .0887 .1230	0.1400 .1427 .1506 .1633 .1603	-1.0000 9294 3589 7866 7166	0.3527 .3527 .3523 .3510 .3485		
1.0 1.2 1.4 1.6 1.8	.0754 .1118 .1571 .2125 .2767	.1611 .2036 .2509 .3032 .3603	.2010 .2243 .2491 .2738 .2963	6493 5811 5143 4496 3876	.3443 .3379 .3290 .5171 .3020		
2.0 2.2 2.4 2.6 2.8	.3569 .4476 .5513 .6683 .7984	.4215 .4859 .5519 .6180	.3149 .3273 .3518 .3270 .3125	3291 2745 2246 1600 1409	.2834 .2616 .2368 .2096 .1811		
3.0 3.2 3.4 3.6 3.8	.9409 1.0950 1.2593 1.4325 1.6130	.7423 .7971 .8450 .8855 .9182	.2888 .2576 .2213 .1827 .1448	1076 0799 0577 0405 0275	.1522 .1242 .0982 .0750 .0553		
4.0 4.2 4.4 4.6 4.8	1.7993 1.9900 2.1839 2.3801 2.5778	.9436 .9625 .9761 .9853 .9913	.1101 .0802 .0559 .0373 .0239	0181 0115 0071 0043	.0393 .0269 .0177 .0112		
5.0 5.2 5.4 5.6 5.8	2.7765 2.9758 3.1754 3.3752 3.5752	.9951 .9974 .9987 .9994 .9998	.0146 .0086 .0048 .0026 .0013	0014 0008 0005 0003 0002	.0040 .0023 .0012 .0066		
6.0	3.7751 3.9751	1.0000 1.0000	.0006 .0003	0002 0001	\$000.		

,		β = -0.36	$S_{w} = -1.0$		
			f"	S	S'
η	f	f'		-1.0000	0.0400
0 .2033 .4024 .5942 .7776	0 .0050 .0199 .0438	0 050 100 150 200	0.2 4 4 8 .2 4 7 6 .2 5 5 2 .2 6 6 3 .2 7 9 3	-1.0000 9186 8391 7628 6905	.0399 .0398 .0398
.9524 1.1193 1.2795 1.4344 1.5555	.1151 .1610 .2130 .2711 .3353	.250 .300 .350 .400 .450	.2929 .3061 .3178 .3274 .3341	6225 5593 4936 4431	.0385 .03765 .03553 .033
1.7 3 4 2 1.8 8 2 4 2.0 3 1 8 2.1 8 4 7 2.3 4 3 9	408397 486653 566528	.500 .550 .600 .550 .700	.3375 .3368 .3317 .3215 .3057	3428 2970 2541 2138 1760	.0318 .0398 .0395 .0351 .023
2.5134 2.6995 2.9129	.8957 1.0400 1.2162	.750 .800 .850	.2833 ,2535 ,2146	1405 1074 0765	.0194 .0162 .0137
3.1 7 6 6. 3.2 3 9 7 3.3 0 7 8 3.3 8 2 5 3.4 6 5 4	1.4473 1.5043 1.5667 1.6358 1.7133	900 910 920 930 940	.1646 .1529 .1406 .1275 .1136	0479 0425 0372 0370	.0089 .0081 .0073 .0065
3.5 5 9 6 3.6 1 2 3 3.6 6 9 7 3.7 5 3 2 3.8 0 4 4 3.8 8 5 9 3.9 8 2 2 4.1 0 1 2	1.8023 1.8525 1.9075 1.9686 2.0375 2.1168 2.2110 2.3279	05050505 55067788 9999999	.0988 .0911 .0830 .0746 .0658 .0472 .0370	0220 0196 0172 0145 01033 0080	.0048 .0043 .0034 .00334 .0035 .00025
4.2 6 0 4 4.3 0 0 3 4.3 4 4 3 4.3 9 3 3 4.4 4 8 9	2.4851 2.5247 2.5682 2.6169 2.6722	01234 99999 99999	.0262 .0239 .0216 .0192 .0167	0038 0034 0030 0026 0022	.0010 .0009 .0008 .0007 .0006
4.5 1 3 4 4.5 9 0 5 4.6 8 7 2 4.8 1 8 7 5.0 3 2 7	2.7 3 6 3 2.8 1 3 0 2.9 0 9 4 3.0 4 0 6 3.2 5 4 3	995 996 9998 9999	.0142 .0117 .0090 .0062	0 0 1 8 0 0 1 4 0 0 1 0 0 0 0 6 0 0 0 3	.0005 .0004 .0003 .0002
6.1 8 4 7	4.3 4 4 6	1,000	٥٥٥٥	.0000	.0000

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		$\beta = -0.3,$	S _w = -1.0		
η	f	f'	f''	S	S'
0 .1 5 6 8 .3 1 2 3 .4 6 5 5 .6 1 5 9	0 .0039 .0155 .0347	.050 .100 .150	0.3181 .3196 .3236 .3292 .3359	-10000 9331 8669 8018 7383	0.4 2 62 .4 2 61 .4 2 55 .4 2 39 .4 2 0 9
.7632 .9077 10498 11902 1.3296	.0941 .1338 .1800 .2326	.250 .300 .350 .400 .450	.3 4 2 7 .3 4 9 1 .3 5 4 3 .3 5 7 9 .3 5 9 1	6766 6169 5593 5039 4506	.4163 .4095 .4005 .3892 .3752
1.4 6 9 0 1.6 0 9 8 1.7 5 3 3 1.9 0 1 6 2.0 5 7 2	.3581 .4320 .5146 .6072 .7123	.500 .550 .600 .650	.3575 .3525 .3436 .3302 .3117	3994 3502 3031 2580 2148	.3586 .3393 .3170 .2918 .2633
2.2 2 4 0 2.4 0 8 0 2.6 2 0 2	.8333 .9761 1.1513	.750 .800 .850	.2871 .2555 .2155	1735 1342 0968	.2315 .1960 .1565
2.8 8 3 5 2.9 4 6 5 3.0 1 4 7 3.0 8 9 5 3.1 7 2 6	1.3819 1.4389 1.5014 1.5705 1.6483	.900 .910 .920 .930 .940	.1646 .1528 .1404 .1273 .1134	0616 0549 0482 0417 0352	.1122 .1026 .0928 .0828 .0724
3.2670 3.3199 3.3775 3.4412 3.5127 3.5945 3.6912 3.8108	1.7375 1.7878 1.8430 1.9043 1.9735 2.0531 2.1477 2.2652	99999999999999999999999999999999999999	.0986 .0908 .0827 .0744 .0657 .0565 .0470	0289 0258 0227 0197 0167 0137 0108 0080	.0617 .0562 .0506 .0449 .03391 .03371 .0208
3.9 7 0 8 4.0 1 0 9 4.0 5 5 1 4.1 0 4 4 4.1 6 0 3	2.4 2 3 2 2.4 6 2 9 2.5 0 6 7 2.5 5 5 6 2.6 1 1 2	.990 .991 .992 .993 .994	.0260 .0238 .0214 .0191 .0167	0052 0046 0041 0035 0030	.0143 .0129 .0116 .0103
4.2 2 5 1 4.3 0 2 6 4.3 9 9 7 4.5 3 1 9 4.7 4 7 1	2.6756 2.7528 2.8495 2.9814 3.1963	.995 .996 .998 .999	.0142 .0116 .0090 .0062	0025 0020 0015 0009 0004	.0075 .0061 .0046 .0031
5.8 5 4 2	4.3031	1.000	.0000	0000	.0000

		$\beta = -0.14$	S _w = -1.0	- / ·	
η	f	f'	· f 11	S .	S'
0 .1199 .2397 .3591 .4783	0 .0029 .0119 .0269 .0477	0 .050 .100 .150 .200	0.4165 .4170 .4179 .4190 .4199	0 9 4 5 3 8 9 0 8 8 3 6 5 7 8 2 5	0.4554 .4554 .4551 .4541 .4522
.5973 .7164 .8359 .9563 1.0782	.0745 .1072 .1461 .1912 .2431	.250 .300 .350 .400	.4202 .4194 .4173 .4133 .4072	7288 6756 6229 5708 5192	.4490 .4443 .4377 .4290 .4179
1.2023 1.3296 1.4615 1.5995 1.7462	.3020 .36.89 .4448 .5311 .6302	.500 .550 .600 .650	.3986 .3869 .3717 .3525 .3287	4682 4178 3681 3191 2707	.4040 .3872 .3671 .3433 .3153
1.9053 2.0829 2.2897	.7 4 5 7 .8 8 3 4 1.0 5 4 3	.750 .800 .850	2995 2639 2204	2 2 3 2 1 7 6 4 1 3 0 4	.2827 .2447 .2005
2.5 4 8 4 2.6 1 0 6 2.6 7 8 1 2.7 5 2 3 2.8 3 4 8	1.2809 1.3373 1.3991 1.4677 1.5448	900 910 920 930 940	.1669 .1547 .1419 .1284 .1142	0855 0766 0678 0591 0504	.1483 .1367 .1247 .1121 .0990
2.9287 2.9813 3.0387 3.1023 3.1738 3.2555 3.3521 3.4717	1.6 3 3 6 1.6 8 3 7 1.7 3 8 7 1.7 9 9 9 1.8 6 9 1 1.9 4 8 6 2.0 4 3 0 2.1 6 0 5	95505 99665 997788 9989	.09912 .0912 .0831 .0746 .06566 .0470 .0369	0417 0374 0331 0289 02404 0163 0121	.0853 .0782 .0709 .0634 .0556 .0476 .0392
3.6319 3.6721 3.7163 3.7657 3.8217	2.3187 2.3585 2.4024 2.4514 2.5071	.990 .991 .992 .993 .994	.0260 .0237 .0214 .0190 .0166	0080 0072 0063 0055 0047	.0213 .0194 .0174 .0155 .0135
3.8867 3.9643 4.0616 4.1939 4.4088	2.5 7 1 7 2.6 4 9 0 2.7 4 5 9 2.8 7 7 8 3.0 9 2 4	.995 .996 .997 .998 .999	0142 0116 0090 0062 0033	-0039 -0031 -0023 -0015 -0007	.0114 .0093 .0071 .0049
5.4938	4.1772	1000	.0000	٥٥٥٥	.0000

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		$\beta = 0.5$,			
η	f	f'	f''	S	S1
0 .0861 .1726 .2596 .3474	0 .0021 .0086 .0195	0 .050 .100 .150 .200	0.5 8 0 6 .5 7 9 7 .5 7 7 0 .5 7 2 4 .5 6 5 9	-1.0000 9574 9147 8718 8285	0.4 9 4 8 .4 9 4 8 .4 9 4 6 .4 9 4 0 .4 9 2 9
.4364 .5270 .6195 .7145	.0549 .0798 .1099 .1456	.250 .300 .350 .400	.5 5 7 4 .5 4 6 8 .5 3 4 0 .5 1 8 8 .5 0 1 1	7848 7406 6957 6501 6037	.4910 .4881 .4839 .4781 .4704
.9145 1.0211 1.1337 1.2539 1.3842	2358 2918 3566 4318 519	.5500 .6500 .700	.4808 .4576 .4310 .4008	5564 5081 4587 4081 3561	.4605 .4479 .4319 .4120
1.5283 1.6922 1.8866	.6244 .7516 .9122	.750 .800 .850	.3277 .2834 .2324	3026 2474 1903	.3567 .3189 .2715
2.1344 2.1947 2.2603 2.3327 2.4135	1.1 2 9 3 1.1 8 3 9 1.2 4 4 0 1.3 1 0 9 1.3 8 6 5	.900 .910 .920 .930 .940	.1727 .1595 .1457 .1313 .1163	1307 1185 1061 0936 0809	.2110 .1968 .1817 .1658 .1486
2.5 0 5 9 2.5 5 7 8 2.6 1 4 6 2.6 7 7 6 2.7 4 8 6 2.8 3 0 0 2.9 2 6 5 3.0 4 6 2	1.4 7 3 8 1.5 2 3 3 1.5 7 7 7 1.6 3 8 4 1.7 0 7 1 1.7 8 6 2 1.8 8 0 5 1.9 9 8 1	99999999999	.1005 .0923 .0839 .0752 .0668 .0470	-06816 -065551 -04816 -04815 -04811 -02815	.1303 .1205 .0719 .0998 .0887 .0769 .0644
3.2069 3.2473 3.2918 3.3415 3.3980	2.1 5 6 8 2.1 9 6 9 2.2 4 1 0 2.2 9 0 4 2.3 4 6 4	990 9992 993 994	.0259 .0236 .0213 .0189	-0145 -0131 -0117 -0102 -0088	.0366 .0335 .0303 .0271 .0238
3.4635 3.5418 3.6400 3.7738 3.9916	2.4 1 1 6 2.4 8 9 5 2.5 8 7 4 2.7 2 0 9 2.9 3 8 4	995 996 998 999	.0140 .0115 .0089 .0061	-0074 -0059 -0044 -0030 -0015	.0203 .0168 .0131 .0092
5.1056	4.0521	1000	.0000	0000-	.0000

		$\beta = 2.0$,	S _w = -1.0		
· 7)	f	f'	f''	S	S'
0.0678 .1360 .2051 .2754	0 .0016 .0068 .0154 .0277	0 .050 .100 .150	0.7381 .7359 .7293 .7188 .7045	-1.0000 9647 9292 8933 8567	0.5203 .5203 .5201 .5198 .5190
3472 4212 4976 5772	.0439 .0643 .0892 .1191 .1545	.2.50 .300 .350 .400	.6869 .6660 .6420 .6150 .5852	8196 7814 7421 7012 6593	.5177 .5157 .5128 .5086 .5029
.7485 .8421 .9427 1.0520 1.1726	.1.964 .2456 .3035 .3719 .4534	.5 0 0 .5 5 0 .6 0 0 .6 5 0 .7 0 0	.5525 .5170 .4786 .4372 .3925	6155 - 5696 - 5215 4707 4172	.4953 .4852 .4721 .4551 .4330
1.3085 1.4659 1.6562	.5520 .6742 .8314	.750 .800 .850	.3 4 4 4 .2 9 2 4 .2 3 5 5	- 3603 - 2996 - 2344	.4045 .3674 .3187
19030 19636 20299 21031 21852	1.0 4 7 7 1.1 0 2 6 1.1 6 3 2 1.2 3 1 0 1.3 0 7 8	.900 .910 .920 .930 .940	.1720 .1683 .1442 .1296 .1144	1639 1490 1339 1185 1028	.2528 .2369 .2198 .2014 .1814
2.2793 2.33203 2.339048 2.5275 2.61099 2.7099 2.8327	13967 14471 15027 15648 16351 17163 18130 19338	99995 99999999999999999999999999999999	.0986 .0904 .0821 .0735 .06554 .0458	-0868 -0786 -0704 -0620 -0536 -0450 -0364 -0275	.1598 .1482 .1360 .1232 .1097 .0954 .0801
2.9977 3.0392 3.0849 3.1359 3.1939	20967 21378 21831 22338 22913	990 991 992 993 994	.0252 .0230 .0207 .0184 .0161	-0186 -0167 -0149 -0131 -0113	.0456 .0418 .0379 .0338 .0297
3.2611 3.3414 3.4421 3.5792 3.8023	2.3 5 8 1 2.4 3 8 5 2.5 3 8 5 2.6 7 5 3 2.8 9 8 0	567 99999 99999	.0137 .0112 .0087 .0060	-0094 -0075 -0057 -0038 -0019	.0254 .0209 .0163 .0114
4.9344	4.0 2 9 8	1000	.0000	0000	.0000

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.10, S_W = -0.8$						
η	f	f'	f"	S	S'		
0	0	0	-0.0 6 8 6	-0.8000	0.0 4 4 7		
.4	0053	0258	0 6 0 3	7821	.0 4 4 8		
.8	0202	0482	0 5 1 5	7642	.0 4 5 0		
1.5	0433	0670	0 4 2 5	7461	.0 4 5 5		
1.6	0733	0821	0 3 3 0	7277	.0 4 6 6		
2.0	1085	0933	0 2 3 1	7087	.0 4 8 3		
2.4 2.8 3.2 3.6 4.0	1474 1882 2291 2678 3016	1005 1030 1004 0918 0759	0123 0003 .0136 .0301 .0501	6889 6679 6452 6204 5928	.0509 .0544 .0591 .0653		
4.4	3273	0511	.0746	5616	.0850		
4.8	3410	0154	.1048	5260	.0950		
5.6	3379	.0336	.1413	4853	.1088		
5.0	3121	.0984	.1837	4388	.1241		
6.0	2568	.1811	.2301	3861	.1392		
6.4	1647	.2823	.2752	3278	.1516		
6.8	0287	.3999	.3103	2656	.1578		
7.2	.1566	.5278	.3245	2029	.1541		
7.6	.3935	.6556	.3091	1440	.1383		
8.0	.6794	.7709	.2629	0937	.1118		
6.4 8.8 9.2 9.6 10.0	1.0071 1.3662 1.7457 2.1369 2.5335	.8632 9276 9665 9866 9954	.1964 .1270 .0703 .0332 .0133	0553 0295 0145 0070 0038	.0799 .0497 .0267 .0123		
10.4	2.9324	9987	.0045	0026	.0016		
10.8	3.3321	9997	.0013	0022	.0005		
11.2	3.7321	10000	.0003	0021	.0001		

		$\beta = -0.2685$	$S_{W} = -0.8$		
η	f	f'	f"	S	S١
0 .2 .4 .6 .8	0 0009 0034 0068 0106	0 0089 0152 0186 0187	-00500 0383 0246 0090 .0086	-0.8000 7634 7268 6902 6535	0.1829 .1829 .1830 .1832 .1835
1.0 1.2 1.4 1.6 1.6	0140 0163 0166 0139 0073	0150 0072 .0051 .0224 .0450	.0282 .0499 .0737 .0994 .1272	6168 5799 5430 5059 4687	.1840 .1845 .1851 .1857 .1861
2.0 2.4 2.4 2.6 2.8	.0044 .0224 .0479 .0822 .1264	.0733 .1077 .1483 .1951 .2479	.1566 .1872 .2184 .2493 .2786	4315 3943 3572 3206 2845	.1862 .1857 .1844 .1821 .1783
3.0 3.2 3.4 3.6 3.8	.1817 .2492 .3297 .4239 .5320	.3063 .3695 .4364 .5055 .5750	.3049 .3264 .3413 .3481 .3455	2493 2154 1832 1530 1253	.1730 .1657 .1564 .1451 .1319
4.0 4.2 4.4 4.6 4.8	.6538 .7890 .9365 1.0953 1.2638	.6430 .7076 .7669 .8195 .8644	.3330 .3109 .2807 .2446 .2052	1003 0784 0598 0442 0318	.1172 .1015 .0854 .0697
5.0 5.2 5.4 5.6 5.8	1.4 4 0 5 1.6 2 3 9 1.8 1 2 4 2.0 0 4 7 2.1 9 9 8	.9015 .9309 .9532 .9694 .9808	.1656 .1283 .0954 .0681 .0465	0221 0148 0096 0059	.0420 .0309 .0219 .0150
6.0 6.2 6.4 6.6 6.8	2.3 9 6 8 2.5 9 5 0 2.7 9 4 0 2.9 9 3 5 3.1 9 3 2	.9884 .9933 .9963 .9981 .9991	.0305 .0192 .0116 .0067	0019 0009 0003 .0000	.0062 .0038 .0022 .0012
7.0 7.2	3.3 9 3 1 3.5 9 3 1	.9997 1.0000	.0020	.0003	.0003

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.3088, S_W = -0.8$					
η	f	f'	f"	S	S'	
0 .2 .4 .6	0 .0001 .0007 .0026	0 .0013 .0057 .0136	0 .0137 .0303 .0496	-0.8000 7548 7096 6644 6192	0.2 2 6 1 .2 2 6 0 .2 2 6 0 .2 2 6 8	
1.0 1.2 1.4 1.6 1.8	.0132 .0238 .0393 .0610	.0 4 2 5 .0 6 4 4 .0 9 2 0 .1 2 5 6 .1 6 5 4	.0963 .1234 .1527 .1835 .2151	5741 5209 4843 4399 3960	.2253 .2245 .2231 .2209 .2176	
2.0 2.2 2.4 2.6 2.8	.1276 .1750 .2335 .3041 .3877	.2116 .2639 .3219 .3846 .4514	.2 4 6 5 2 7 6 4 .3 0 3 1 .3 2 4 7 .3 3 9 5	3529 3109 2704 2317 1953	.2130 .2067 .1984 .1881 .1756	
3.0 3.2 3.4 3.6 3.8	.4848 .5957 .7203 .8580 1.0079	.5201 .5890 .6563 .7199	.3458 .3423 .3286 .3055 .2742	1616 1311 1039 0804 0606	.1609 .1445 .1267 .1082 .0898	
4.0 4.2 4.4 4.6 4.6	1.1687 1.3391 1.5173 1.7018 1.8912	.8291 .8727 .9082 .9360 .9570	.2372 .1975 .1580 .1213	0 4 4 4 0 3 1 6 0 2 1 8 0 1 4 5 0 0 9 4	.0723 .0563 .0423 .0306 .0214	
5.0 5.2 5.4 5.6 5.8	2.0842 2.2793 2.4771 2.6754 2.8746	.9722 .9827 .9896 .9940	.0630 ;0426 .0276 .0172 .0103	0058 0035 0020 0011 0006	.0144 .0093 .0058 .0034	
6.0 6.4 6.6 6.8	3.0741 3.2738 3.4737 3.6737 3.8737	.9983 .9996 .9999 1.0000	.0059 .0032 .0017 .0009	0003 0001 0000 .0000	.0011 .0006 .0003 .0001	

		$\beta = -0.325$	$S_{W} = -0.8$		
$-\frac{1}{\eta}$	f	f'	f"	S	S'
0 .2 .4 .6 .8	0 .0011 .0047 .0117 .0227	0 .0113 .0258 .0442 .0672	0.0 493 .0 640 .0 619 .1 0 29 .1 2 69	-0.3250 7491 6982 6474 5966	0.2 5 4 5 .2 5 4 5 .2 5 4 4 .2 5 4 0 .2 5 3 1
1.0 1.2 1.4 1.6 1.8	.0389 .0611 .0907 .1288 .1765	.0952 .1287 .1681 .2135 .2649	.1535 .1821 .2121 .2423 .2714	5462 4961 4466 3980 3507	.2516 .2491 .2454 .2401 .2329
2.0 2.2 2.4 2.6 2.8	.2351 .3056 .3889 .4855 .5958	.3219 .3837 .4493 .5172 .5855	.2978 .3196 .3349 .3420 .3396	3050 2614 2204 1825 1481	.2236 .2118 .1977 .1812 .1626
3.0 3.2 3.4 3.6 3.8	.7196 .8565 1.0055 1.1656 1.3352	.6523 .7157 .7736 .8253	.3272 .3052 .2749 .2388 .1996	1176 0911 0688 0506, 0361	.1426 .1218 .1012 .0815 .0635
4.0 4.2 4.4 4.6 4.8	1.5127 1.6967 1.8857 2.0783 2.2735	.9052 .9335 .9549 .9705 .9612	.1604 .1236 .0914 .0648 .0440	-0250 -0168 -0110 -0070 -0043	.0477 .0346 .0242 .0163
5.0 5.2 5.4 5.6 5.8	2.4705 2.6687 2.8676 3.0670 3.2666	9830 9959 9975 9984	.0287 .0179 .0108 .0062	- 0027 - 0016 - 0010 - 0007 - 0005	.0065 .0039 .0023 .0013
6.0	3.4664	.9991	.0024	- 2004	.0005

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	•	$\beta = -0.3285$	$s_{\rm w} = -0.8$	_	
η	f	f'	f"	S	s'
0 2 4 6 8	0 .0015 .0063 .0153	0 .0153 .0339 .0565	0.0693 .0842 .1024 .1238 .1482	-0.8000 7471 6943 6415 5888	0.2644 .2644 .2642 .2636 .2625
1.0 1.2 1.4 1.6 1.8	.0491 .0760 .1110 .1553 .2101	.1159 .1538 .1974 .2470 .3021	.1750 .2036 .2331 .2621 .2892	5365 4847 4337 3838 3355	.2605 .2573 .2525 .2459 .2371
20 22 24 26 28	.2765 .3554 .4474 .5530	.3624 .4267 .4939 .5622 .6297	.3125 .3301 .3401 .3412 .3324	2892 2453 2045 1671 1337	.2259 .2121 .1958 .1772 .1568
3.0 3.2 3.4 3.6 3.8	.8047 .9497 1.1061 1.2727 1.4477	.6945 .7546 .8085 .8552 .8939	.3138 .2863 .2520 .2136 .1741	1045 0796 0590 0425 0297	.1353 .1136 .0925 .0729 .0556
4.0 4.2 4.4 4.6 4.8	1.6297 1.8172 2.0088 2.2033 2.3999	.9249 .9487 .9662 .9785 .9868	.1362 .1022 .0735 .0507 .0335	0201 0131 0083 0051 0030	.0408 .0289 .0197 .0129
5.0 5.2 5.4 5.6 5.8	2.5 9 7 8 2.7 9 6 7 2.9 9 6 0 3.1 9 5 6 3.3 9 5 4	.9922 .9956 .9976 .9987	.0212 .0129 .0075 .0042 .0023	0017 0010 0005 0003 0001	.0050 .0029 .0016 .0009
6.0 6.2 6.4 6.6 6.8	3.5954 3.7953 3.9953 4.1953 4.3953	.9997 .9999 .9999 1.0000	.0012 .0006 .0003 .0001	0001 .0000 .0000 .0000	.0002
7.0 7.2	4.5953 4.7953	10000	.0000	.0000	.0000

	$\beta = -0.3285, S_{W} = -0.8$								
$-\overline{\eta}$	f	f'	f"	S	S				
0 24 6 8	.0023 .0096 .0226	0 .0234 .0502 .0810 .1164	0.1100 .1250 .1434 .1650 .1893	-0.8000 7436 6873 6311 5751	0.2818 .2818 .2815 .2806 .2788				
1.0 1.2 1.4 1.6 1.8	.0696 .1054 .1510 .2074 .2755	.1569 .2027 .2540 .3105 .3716	2156 2428 2698 2948 3162	5196 4649 4114 3594 3096	.2758 .2710 .2642 .2549 .2430				
2.0 2.4 2.6 2.8	.3563 .4503 .5579 .6791 .8134	.4366 .5039 .5720 .6391 .7031	.3319 .3401 .3395 .3292 .3093	2624 2185 1784 1425 1111	.2281 .2105 .1904 .1683 .1450				
3.0 3.2 3.4 3.6 3.8	.9600 1.1178 1.2855 1.4615 1.6443	.7622 .8151 .8605 .8981 .9281	.2810 .2463 .2079 .1687 .1314	0845 0625 0449 0313 0211	.1215 .0987 .0776 .0590 .0432				
4.0 4.2 4.4 4.6 4.8	1.8 3 2 3 2.0 2 4 2 2.2 1 9 0 2.4 1 5 8 2.6 1 3 8	.9510 .9678 .9795 .9875 .9926	.0982 .0704 .0483 .0318	0138 0087 0053 0032 0018	.0305 .0207 .0136 .0085				
5.0 5.2 5.4 5.6 5.8	2.8 1 2 7 3.0 1 2 1 3.2 1 1 7 3.4 1 1 5 3.6 1 1 4	.9957 .9976 .9987 .9993 .9996	.0121 .0071 .0039 .0021	0 0 1 0 0 0 0 5 0 0 0 3 0 0 0 2 0 0 0 1	.0030 .0017 .0009 .0005				
6.0 6.2	3.8 1 1 3 4.0 1 1 3	.9998 .9999	•0008 •0006	0001 .0000	.0001				

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

			25, S _w = -0			<u> </u>
η	<u>f</u>	f'	f''	S	S'	_
0 .33.79 .6210 .8632 10761	.0080 .0289 .0590	.050 .100 .150 .200	0.1 3 5 3 .1 6 2 3 .1 9 2 2 .2 2 1 4 .2 4 8 7	-0.8000 7015 6192 5494 4888	0.29 13 .29 11 .28 97 .28 66 .28 19	0
1.2 6 7 7 1.4 4 3 9 1.6 0 8 7 1.7 6 5 4 1.9 1 6 5	.1391 .1875 .2410 .2997 .3639	.250 .300 .350 .400	.2732 .2944 .3120 .3258 .3354	4353 3874 3440 3042 2677	.27 57 .26 79 .25 87 .24 79 .23 59	1 1 1 1
2.0 6 4 3 2.2 1 0 8 2.3 5 8 2 2.5 0 8 8 2.6 6 5 6	.4341 .5110 .5958 .6899 .7958	.500 .550 .600 .650	.3 4 0 6 .3 4 1 2 .3 3 6 6 .3 2 6 5 .3 1 0 3	-2338 -2023 -1729 -1454 -1197	.2224 .2075 .1913 .1737 .1545	1 1 2 2 2
2.8 3 2 7 3.0 1 6 1 3.2 2 6 8	9170 1.0592 1.2332	.750 .800 .850	2874 2569 2172	0956 0731 0521	.1339 .1118 .0878	3 3 3
3.4876 3.5500 3.6175 3.6915 3.7737	1.4618 1.5182 1.5800 1.6484 1.7253	.900 .910 .920 .930 .940	.1663 .1544 .1419 .1286 .1146	0328 0291 0255 0219 0185	.0618 .0563 .0507 .0450 .0392	3 3 3 3 3
3.8672 3.9194 3.9764 4.0394 4.1101 4.1911 4.2867 4.4050	1.8136 1.8134 1.9180 1.9786 2.0470 2.1258 2.2192 2.3354	99999999999999999999999999999999999999	.09918 .0918 .0836 .07564 .0578 .0475	0151 0134 0118 0102 0086 0071 0056 0041	.0332 .0302 .0271 .0239 .0208 .0176 .0143	3 3 3 3 3 3 3 4
4.5 6 3 2 4.6 0 2 9 4.6 4 6 6 4.6 9 5 3 4.7 5 0 7	2.4917 2.5310 2.5743 2.6228 2.6777	.990 .991 .992 .993	.0263 .0240 .0217 .0193	0026 0024 0021 0018 0015	.0 0 74 .0 0 67 .0 0 6.0 .0 0 53 .0 0 4 6	4 4 4
4.8148 4.8915 4.9876 5.1186 5.3319	2.7415 2.8178 2.9136 3.0443 3.2573	.995 .997 .999 .999	.0143 .0117 .0091 .0063	0013 0010 0007 0005 0002	.0038 .0031 .0024 .0016	4 4 4 4 5
6.4255	4.3506	1.000	.0 0 00	0000	.0000	6

$\beta = -0.3$, $S_W = -0.8$							
η .	f	f'	f!!	S	S١		
.2310 .4447 .6423 .8259	0. .0056 .0215 .0461 .0782	0 .050 .100 .150 .200	0.2086 .2248 .2435 .2629 .2819	-0.8000 7271 6597 5978 5407	0.3154 .3154 .3146 .3126 .3091		
.9979 .1605 .3159 .4660	.1 1 6 8 .1 6 1 5 .2 1 2 0 .2 6 8 2 .3 3 0 4	.250 .300 .350 .400	.2995 .3151 .3280 .3380 .3445	4879 4390 3934 3507 3108	.3040 .297 .288 .278		
.7568 .9010 .0468 .1964	.3990 .4747 .5586 .6521 .7576	.5 0 0 .5 5 0 .6 0 0 .6 5 0	.3472 .3457 .3395 .3282	2732 2378 2044 1729 1431	.2537 .237 .220 .201		
.5194 .7030 .9142	.8786 1.0210 1.1954	.750 .800 .850	2875 2565 2166	1149 0883 0633	.157 .132 .104		
.1759 .2385 .3063 .3806 .4631	1.4 2 4 6 1.4 8 1 3 1.5 4 3 3 1.6 1 2 0 1.6 8 9 2	.900 .910 .920 .930	.1657 .1538 .1413 .1281 .1141	0 4 0 0 0 3 5 6 0 3 1 2 0 2 6 9 0 2 2 7	.074 .067 .061 .054		
5569 6094 .6667 .7299 .8010 .8823 .9784	1.7 7 7 9 1.8 2 7 9 1.8 8 2 7 1.9 4 3 6 2.0 1 2 3 2.0 9 1 5 2.1 8 5 4 2.3 0 2 2	.955 .955 .965 .975 .985	.0992 .0914 .0833 .0749 .0669 .0473	0186 0166 0146 -:0126 0107 0088 0069 0051	.040 .036 .033 .029 .025 .021 .017		
.2562 .2961 .3400 .3890	2.4592 2.4986 2.5422 2.5908 2.6460	.990 .991 .992 .993	.0262 .0239 .0216 .0192	0033 0029 0026 0022 0019	.009 .008 .007 .006		
.5090 .5860 .6827 .8143	2.7 1 0 1 2.7 8 6 8 2.6 8 3 1 3.0 1 4 4 3.2 2 8 5	.995 .996 .997 .998 .999	.0142 .0117 .0090 .0062	0016 0012 0009 0006 0003	.004 .003 .002 .002		

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		β = -0.14	$S_{w} = -0.8$		
η	f	f'	f''	S	S'
0 .1294 .2575 .3841 .5094	0 .0032 .0128 .0286 .0505	0 .050 .100 .150	0.38 41 .38 82 .39 26 .39 70 .40 09	-0.8000 7535 7075 6622 6174	0.3590 .3590 .3590 .3580 .3560
.6337 .7572 .8804 1.0039	.0784 .1124 .1524 .1987 .2516	.250 .300 .350 .400	.4039 .4057 .4058 .4039 .3996	5734 5300 4873 4453 4040	.3530 .3490 .3440 .3360 .3270
1.2546 1.3837 1.5170 1.6561 1.8038	.3116 .3795 .4562 .5432 .6429	.5 0 0 .5 5 0 .6 0 0 .6 5 0 .7 0 0	.3925 .3822 .3682 .3500 .3269	3634 3235 2843 2459 2081	.3160 .3020 .2860 .2670 .2440
1.9636 2.1417 2.3489	.7589 .8971 1.0682	.750 .800 .850	.2984 .2633 .2202	1711 1349 0995	.2190 .1890 .1540
2.6077 2.6699 2.7375 2.8116 2.8941	1.2 9 5 0 1.3 5 1 3 1.4 1 3 1 1.4 8 1 7 1.5 5 8 8	.900 .910 .920 .930 .940	.1669 .1547 .1419 .1285 .1143	0650 0582 0515 0448 0381	.1140 .1050 .0950 .0860 .0760
29879 30405 304079 31614 32328 33145 34110 35305	1.6475 1.6976 1.7526 1.8138 1.8823 2.0566 2.1741	99999999999999999999999999999999999999	.0992 .0913 .0831 .0747 .0659 .0567 .0471	0316 0283 0250 0218 0154 0122 0091	.0650 .0650 .0540 .0480 .0420 .0360 .0330
36905 37306 37749 38243 38803	2.3 3 2 0 2.3 7 1 8 2.4 1 5 8 2.4 6 4 7 2.5 2 0 4	990 991 992 993 994	.0260 .0237 .0214 .0191	0060 0054 0047 0041 0035	.0160 .0150 .0130 .0120
39452 40228 41201 42524 44678	2,5849 2,66,22 2,7591 2,8912 3,1062	995 996 997 998 999	.0142 .0116 .0090 .0062	0029 0023 0017 0011 0005	.0090 .0070 .0050 .0040
55574	41955	1000	.0000	.0000	.0000

		β = 0.5,	S _w = -0.8		
η	f	f'	f''	S	Si
0 0768 1547 2339 3147	0 .0019 .0077 .0177 .0318	0 .050 .100 .150 .200	0.6546 .6464 .6368 .6258 .6133	-08000 -7690 -7375 -7056 -6731	0.4 0 3 .4 0 3 .4 0 3 .4 0 3 .4 0 2
.3971 .4817 .5687 .6586 .7520	.0504 .0737 .1020 .1357 .1755	.250 .300 .350 .400 .450	.5993 .5835 .5659 .5463 .5246	6400 6063 5717 5363 5000	.401 .399 .396 .391
8496 9524 10614 11784 13058	.2219 .2759 .3387 .4119 .4979	.500 .550 .600 .650	.5 0 0 4 .4 7 3 7 .4 4 4 1 .4 1 1 1 .3 7 4 3	- 4627 - 4243 - 3848 - 3439 - 3016	.379 .369 .357 .342 .323
1.4 4 7 2 1.6 0 8 6 1.8 0 1 0	.6006 .7258 .8848	.750 .800 .850	.3333 .2870 .2345	2577 2119 1641	.298 .268 .230
2.0 4 6 9 2.1 0 6 9 2.1 7 2 3 2.2 4 4 4 2.3 2 5 0	1.1003 1.1546 1.2144 1.2811 1.3565	.900 .910 .920 .930 .940	.1737 .1603 .1463 .1318 .1166	1137 1032 0926 0819 0710	.180 .169 .156 .143
24171 24690 25257 25887 26596 27410 28375 29573	1.4 4 3 6 1.4 9 3 0 1.5 4 7 3 1.6 0 7 9 1.6 7 6 6 1.7 5 5 7 1.8 5 0 1 1.9 6 7 8	99999999999999999999999999999999999999	.1007 .0925 .0840 .0752 .0668 .0470 .0367	0599 0543 0486 0429 0371 0312 0252 0192	.113 .104 .096 .087 .077 .067 .057
31183 31588 32034 32532 33098	2,1268 2,1669 2,2111 2,2606 2,3168	.990 .991 .992 .993	.0258 .0235 .0212 .0189 .0165	- 0130 - 0117 - 0105 - 0092 - 0079	.032 .030 .027 .024
3.3755 3.4540 3.5526 3.6869 3.9056	2.38 2 1 2.4603 2.5586 2.6925 2.9109	.995 .996 .997 .998	.0140 .0115 .0088 .0061	- 0066 - 0053 - 0040 - 0027 - 0013	.018 .015 .012 .008
5.0257	4.0307	1.000	-0 0 00	0000	.000

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		TABLE 1.			
		$\beta = 1.5$,	$S_{\mathbf{W}} = -0.8$		
η	f	f'	f"	S	S¹
0 0581 1176 1788 2418	0 .0014 .0059 .0136 .0246	0 .050 .100 .150 .200	0.8 6 8 9 .8 5 0 4 .8 2 9 6 .8 0 6 5 .7 8 1 2	-0.8 0 0 0 7752 7498 7238 6970	0.4261 4261 4260 4258 4253
.3069 .3746 .4453 .5194	.0393 .0580 .0810 .1088 .1421	.250 .300 .350 .400	.7537 .7240 .6922 .6582	- 6693 - 6406 - 6108 - 5797 - 5471	.4245 .4231 .4211 .4182 .4142
.6805 .7694 .8653 .9701	.1815 .2283 .2836 .3491 .4277	.500 .550 .600 .650	.5838 .5432 .5003 .4547 .4064	5130 4770 4389 3985 3554	.4088 .4015 .3919 .3792 .3625
1.2179 1.3711 1.5572	.5232 .6420 .7958	.750 .800 .850	.3549 .2998 .2402	3092 2592 2049	.3406 .3117 .2729
1.7999 1.8597 1.9252 1.9977 2.0790	1.0085 1.0627 1.1226 1.1897 1.2657	900 910 920 930 940	.1744 .1603 .1458 .1308 .1154	1 452 1 324 1 194 1 061 0 924	.2195 .2063 .1921 .1767 .1599
21724 2250 22857 23469 24193 25025 26013 27240	1.3540 1.4041 1.4594 1.5212 1.5912 1.6721 1.7687 1.8893	955 955 965 975 975 985	.0993 .0910 .0825 .0638 .06555 .0459	0784 0712 0639 0565 0483 0413 0335	.1416 .1317 .1213 .1103 .0986 .0861 .0726 .0580
2.8 8 9 1 2.9 3 0 7 2.9 7 6 4 3.0 2 7 6 3.0 8 5 6	2.0524 2.0935 2.1389 2.1897 2.2473	.990 .991 .992 .993 .994	.0252 .0229 .0207 .0184 .0160	0173 0156 0139 0122 0105	.0419 .0385 .0349 .0313
3.1 5 3 0 3.2 3 3 6 3.3 3 4 7 3.4 7 2 4 3.6 9 6 4	2.3144 2.3946 2.4953 2.6327 2.8564	.995 .996 .997 .998 .999	.0136 .0112 .0086 .0060	0088 0071 0054 0036 0018	.0236 .0195 .0152 .0107
4.8362	3.9959	1.000	.0000	.0000	.0000

$\beta = 2.0, S_W = -0.8$							
	f f'	f" .	S	S'			
η 0 5 3 3 1 0 8 1 1 6 4 5 2 2 2 9	0 0 0 .0013 .050 .0054 .100 .0125 .150 .0227 .200	0.9 4 8 0 .9 2 5 5 .9 0 0 3 .8 7 2 4 .8 4 2 1	-08000 -7768 -7531 -7287 -7035	0.4331 .4331 .4330 .4329 .4324			
2834 3466 4128 4826 5565	.0364 .250 .0538 .300 .0754 .350 .1016 .400 .1331 .450	.8 0 9 4 .7 7 4 3 .7 3 7 0 .6 9 7 5 .6 5 5 8	- 6773 - 6501 - 6216 - 5918 - 5604	.4317 .4305 .4287 .4261 .4225			
.6355 .7205 .8129 .9144 1.0278	.1706 .500 .2153 .550 .2686 .600 .3321 .650 .4087 .700	.6121 .5662 .5181 .4678	- 5273 - 4921 - 4546 - 4144 - 3711	.4175 .4108 .4019 .3898 .3738			
11570 13086 14941	.5025 .750 .6202 .800 .7735 .850	.3600 .3019 .2401	-3242 -2729 -2166	.3525 .3239 .2848			
1.7377 1.7980 18640 1.9372 2.0193	.9870 .900 1.0415 .910 1.1020 .920 1.1697 .930 1.2465 .940	.1732 .1590 .1445 .1296 .1142	-1539 -1405 -1267 -1125 -0980	.2299 .2163 .2016 .1856 .1680			
21137 21670 22254 22903 23636 24477 25476 26718	1.3357 1.3864 1.4424 1.5048 1.5758 1.6576 1.7553 1.8773	.0982 .0900 .0815 .0729 .0649 .0454	0831 0754 0677 05918 0437 0353 0269	.1488 .1384 .1274 .1158 .1035 .0904 .0762			
28387 28806 29269 29785 30371	2.0421 ,990 2.0837 ,991 2.1295 ,992 2.1808 ,993 2.2390 ,994	.0249 .0227 .0205 .0182 .0159	-0182 -0164 -0146 -0129 -0111	.0 4 38 .0 4 02 .0 3 65 .0 3 86			
31051 31864 32883 34270 36526	2,3066 2,3875 2,4891 2,6274 2,8527 2,999	.0135 .0111 .0086 .0059	-0093 -0074 -0056 -0037 -0019	.0246 .0203 .0158 .0111			
4.7958	3.9956 1.000	.0000	0000	.0000			

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.2350$, $S_{W} = -0.4$						
η	f	f'	f"	S	S١		
0 2 4 6 8	0 0008 0025 0038 0035	0 0071 0084 0037 0073	-0.0500 0213 .0085 .0393	-0.4000 3779 3557 3336 3114	0.1 1 0 7 .1 1 0 7 .1 1 0 7 .1 1 0 8 .1 1 0 8		
1.0 1.2 1.4 1.6 1.8	0004 .0069 .0197 .0393	.0249 .0490 .0800 .1179 .1627	.1041 .1379 .1722 .2067 .2404	2893 2671 2449 2229 2010	.1109 .1108 .1106 .1099 .1088		
2.0 2.4 2.6 2.8	.1048 .1533 .2138 .2673 .3745	.2140 .2714 .3342 .4012 .4710	.2724 .3013 .3256 .3434 .3533	1794 1583 1378 1182 0996	.1069 .1042 .1005 .0956 .0895		
3.0 3.2 3.4 3.6 3.8	.4757 .5912 .7203 .8625 1.0165	.5419 .6119 .6791 .7414 .7974	.3539 .3445 .3252 .2970 .2621	0825 0668 0529 0409 0308	.0822 .0739 .0649 .0554		
4.0 4.2 4.4 4.6 4.8	1.1809 1.3543 1.5350 1.7214 1.9122	.8 4 5 9 .8 8 6 5 .9 1 9 0 .9 4 4 2 .9 6 2 8	.2229 .1825 .1436 .1085 .0786	0225 0160 0110 0073 0048	.0369 .0286 .0214 .0155		
5.0 5.2 5.4 5.6 5.8	2.1 0 6 2 2.3 0 2 4 2.5 0 0 0 2.6 9 8 6 2.8 9 7 8	.9760 .9850 .9910 .9947 .9969	.0547 .0365 .0234 .0144 .0085	0030 0018 0011 0006 0003	.0072 .0046 .0029 .0017		
6.0 6.2 6.4 6.6 6.8	3.0973 3.2970 3.4969 3.6968 3.8967	.9982 .9983 .9995 .9997	.0048 .0027 .0014 .0007 .0003	0002 0001 0001 0001	.0005 .0003 .0001 .0001		

	$\beta = -0.2460, S_{W} = -0.4$								
η	f	f!	f"	S	Si				
0 .2 .4 .6	0 .0002 .0016 .0055 .0131	0 .0030 .0121 .0277 .0496	0 .0301 .0615 .0940 .1276	-0.4000 3750 3500 3251 3001	0.1 2 4 9 1 2 4 9 1 2 4 6 1 2 4 6				
1.0 1.2 1.4 1.6 1.8	.0259 .0451 .0721 .1084 .1552	.0787 .1146 .1573 .2067 .2623	1618 1964 2305 2631 2929	2752 2504 2260 2018 1782	.1241 .1233 .1218 .1197 .1166				
2.0 2.2 2.4 2.6 2.8	.2137 .2849 .3696 .4683 .5810	.3235 .3893 .4582 .5285 .5985	.3184 .3379 .3497 .3524 .3452	1552 1333 1126 0933 0758	.1124 .1069 .1002 .0922 .0330				
3.0 3.2 3.4 3.6 3.8	.7075 .8471 .9988 1.1611 1.3326	.6660 .7291 .7861 .8360 .8780	.3280 .3017 .2681 .2298 .1896	0602 0466 0352 0258 0184	.0730 .0625 .0520 .0419 .0327				
4.0 4.2 4.4 4.6 4.8	1.5118 1.6969 1.8867 2.0798 2.2753	.9119 .9384 .9581 .9723	.1504 .1146 .0839 .0589 .0397	0127 0084 0054 0034	.0246 .0178 .0124 .0084				
5.0 5.2 5.4 5.6 5.8	2.4724 2.6705 2.8693 3.0685 3.2679	9885 9952 9957 9975	.0257 .0160 .0096 .0056	0011 0006 0003 0001 .0000	.0034				
6.0	3,4675	.9980	.0018	0000	0002				

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.2483$, $S_W = -0.4$							
η	f	f'	f"	S	S1			
0 .2 .4 .6 .8	.0012 .0056 .0145 .0293	0 .0130 .0323 .0580 .0903	0.0 5 0 0 .0 8 0 5 .1 1 2 2 .1 4 5 0 .1 7 8 6	-0.4000 3728 3456 3184 2913	0.1 3 6 0 .1 3 6 0 .1 3 5 9 .1 3 5 7 .1 3 5 1			
1.0 1.2 1.4 1.6 1.8	.0511 .0615 .1216 .1728 .2362	.1 2 9 4 .1 7 5 2 .2 2 7 4 .2 8 5 6 .3 4 8 9	.2123 .2453 .2765 .3045 .3276	2644 2378 2116 1860 1613	.1340 .1323 .1296 .1259 .1209			
2.0 2.2 2.4 2.6 2.8	.3126 .4028 .5071 .6254 .7572	.4162 .4860 .5565 .6258	.3441 .3524 .3513 .3402 .3193	1377 1156 0952 0768 0605	.1145 .1066 .0973 .0869 .0757			
3.0 3.2 3.4 3.6 3.8	.9018 1.0580 1.2243 1.3992 1.5811	.7530 .8075 .8544 .8933 .9243	2899 2541 2146 1744 1362	0 4 6 5 0 3 4 8 0 2 5 3 0 1 7 9 0 1 2 3	.0641 .0528 .0420 .0323			
4.0 4.2 4.4 4.6 4.8	1.7684 1.9598 2.1542 2.3507 2.5485	.9480 .9654 .9778 .9861 .9915	.1021 .0734 .0506 .0335 .0213	0 0 8 2 0 0 5 4 0 0 3 4 0 0 2 2 0 0 1 4	.0172 .0118 .0078 .0050			
5.0 5.2 5.4 5.6 5.8	2.7471 2.9463 3.1458 3.3455 3.5453	.9949 .9969 .9980 .9987 .9991	.0 1 3 0 .0 0 7 6 .0 0 4 3 .0 0 2 4 .0 0 1 3	0009 0006 0005 0004 0003	.0018 .0010 .0006 .0003			
6.0 6.2 6.4	3.7 4 5 1 3.9 4 5 0 4.1 4 4 8	.9992 .9993 .9994	.0006 .0004 .0000	0 0 0 3 0 0 0 3 0 0 0 3	.0001 .0000 .0000			

		β = -0.24	., S _w = -0.4		
	f	f'	f'	S	S'
7 0 .3766 .6490 .8732 1.0687	0 .0083 .0284 .0562	0 .050 .100 .150	0.1064 .1628 .2056 .2409	- 0.4 0 0 0 - 3 4 4 7 3 0 5 0 2 7 2 6 2 4 4 7	0.1473 .1460 .1452 .1438 .1418
1.2453 1.4087 1.5628 1.7106 1.8541	.1299 .1748 .2248 .2802 .3412	.250 .300 .350 .400 .450	.2956 .3160 .3322 .3442 .3518	2199 1974 1768 1577 1399	.1390 .1356 .1315 .1267 .1212
1.9955 2.1366 2.2793 2.4259 2.5793	.4083 .4824 .5644 .6561 .7596	.500 .550 .600 .650	.3549 .3533 .3466 .3346	1232 1074 0926 0785 0651	.1150 .1080 .1002 .0916 .0822
2.7433 2.9242 3.1328	.8786 1.0189 1.1912	.750 .800 .850	.2921 .2600 .2191	0525 0405 0292	.0719 .0606 .0481
3.3 9 1 8 3.4 5 3 8 3.5 2 1 0 3.5 9 4 7 3.6 7 6 7	1.4 1 8 1 1.4 7 4 2 1.5 3 5 7 1.6 0 3 9 1.6 8 0 6	.900 .910 .920 .930 .940	.1672 .1552 .1425 .1291 .1149	0186 0165 0145 0126 0106	.0343 .0314 .0283 .0252
3.7698 3.8220 3.8789 3.9418 4.0125 4.0934 4.1891 4.3074	1.7686 1.8183 1.8728 1.9333 2.0017 2.0805 2.1740 2.2902	.9 5 0 .9 5 5 .9 6 5 .9 7 0 .9 8 0 .9 8 5	.0999 .0920 .0838 .0753 .06572 .0475	0087 0078 0068 0059 0041 0033 0024	.0188 .0171 .0154 .0136 .0119 .0101 .0082
4.4658 4.5056 4.5493 4.5982 4.6536	2.4 4 6 7 2.4 8 6 1 2.5 2 9 5 2.5 7 8 0 2.6 3 3 0	.991 .992 .993 .994	.0263 .0240 .0216 .0192 .0168	_0016 _0014 _0012 _0011 _0009	.0043 .0039 .0035 .0031
4.7179 4.7947 4.8912 5.0226 5.2317	2.6 9 7 0 2.7 7 3 5 2.8 6 9 6 3.0 0 0 6 3.2 1 4 4	.995 .996 .997 .999	.0143 .0117 .0090 .0062 .0049	-0008 -0006 -0004 -0003 -0001	.0022 .0018 .0014 .0009
6.5567	4.5 4 1 4	1000	.0000	-0000	.0000

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		$\beta = -0.2,$	$S_{W} = -0.4$		
η	f	f'	f''	S	S'
0 .2161 .4108 .5892 .7551	0 .0052 .0197 .0419	0 .050 .100 .150	0.2182 .2447 .2692 .2914 .3111	-0.4 0 0 0 3 6 4 8 3 3 3 2 3 0 4 4 2 7 7 8	0.1626 .1625 .1621 .1612 .1597
.9115 1.0606 1.2044 1.3444 1.4821	.1060 .1470 .1937 .2461 .3047	.250 .300 .350 .400 .450	.3281 .3422 .3531 .3607 .3646	-2529 -2297 -2077 -1869 -1671	.1575 .1546 .1501 .1463 .1409
1.6191 1.7569 1.8971 2.0419 2.1939	.3697 .4421 .5227 .6132 .7159	.500 .550 .600 .650	.3647 .3606 .3519 .3381 .3188	1482 1302 1129 0964 0805	.1345 .1272 .1189 .1096 .0990
2.3 5 7 1 2.5 3 7 5 2.7 4 6 0	.8343 .9742 1.1464	.750 .800 .850	.2932 .2604 .2190	0653 0508 0369	.0873 .0742 .0595
3.0 0 5 4 3.0 6 7 6 3.1 3 5 0 3.2 0 8 9 3.2 9 1 1	1.3737 1.4300 1.4916 1.5600 1.6369	.900 .910 .920 .930 .940	.1668 .1548 .1421 .1287 .1146	- 0236 - 0211 - 0186 - 0161 - 0136	.0429 .0393 .0356 .0318
3.3846 3.4369 3.43641 3.5572 3.6281 3.7094 3.8054 3.9240	1.7253 1.7751 1.8298 1.8906 1.9592 2.0382 2.1321 2.2486	9550 99665 9975 9985 9985	.0995 .0916 .0835 .07562 .0570 .0473	0112 0100 0088 0077 0054 0054 0042 0031	.0239 .0218 .0196 .0175 .0152 .0106
4.0829 41228 41668 4.2158 4.2714	2.4056 2.4451 2.4887 2.5373 2.5926	.990 .991 .992 .993	.0262 .0239 .0216 .0192 .0167	- 0020 - 0018 - 0016 - 0014 - 0012	.0 0 5 7 .0 0 5 1 .0 0 4 6 .0 0 4 1
4.3 3 5 8 4.4 1 2 9 4.5 0 9 6 4.6 4 1 1 4.8 5 4 5	2.6 5 6 7 2.7 3 3 4 2.8 2 9 7 2.9 6 0 9 3.1 7 4 0	.995 .996 .997 .998	.0142 .0117 .0090 .0062	0010 0008 0005 0003 0001	.0029 .0024 .0019 .0013
5,9279	4.2471	1.000	.0000	.0000	.0000

·	$\beta = 0.5, S_W = -0.4$						
η	f	f¹	f''	S	S'		
0 .0637 .1290 .1962 .2654	0 .0016 .0065 .0149	.050 .100 .150 .200	0.7946 .7753 .7551 .7339 .7116	-04000 -3866 -3730 -3590 -3445	.209 .209 .209 .209		
.3368 .4108 .4878 .5681 .6523	.0431 .0635 .0886 .1187 .1546	.250 .300 .350 .400 .450	.6881 .6632 .6369 .6090	-3296 -3143 -2984 -2819 -2648	. 20 6 . 20 6 . 20 2		
.7411 .8354 .9364 1.0456 1.1657	.1968 .2464 .3046 .3729 .4540	.500 .550 .600 .650	.5 4 7 7 .5 1 3 8 .4 7 7 4 .4 3 8 1 .3 9 5 6	-2470 -2284 -2090 -1887 -1673	.199 .195 .190 .183		
1.3001 1.4548 1.6406	.5516 .6716 .8252	.750 .800 .850	.3493 .2984 .2417	-1447 -1207 -0950	.163 .148 .129		
1.8804 1.9391 2.0032 2.0741 2.1535	1.0353 1.0885 1.1471 1.2127 1.2870	.900 .910 .920 .930 .940	.1775 .1635 .1490 .1339 .1183	- 0672 - 0613 - 0554 - 0492 - 0430	.103 .097 .090 .083		
2.2 4 4 5 2.2 9 5 7 2.3 5 1 9 2.4 1 4 3 2.4 8 4 6 2.5 6 5 4 2.6 6 1 4 2.7 8 0 7	1.3730 1.4218 1.4756 1.5356 1.6037 1.6823 1.7761 1.893	999999999999	1020 .0935 .0848 .0759 .0667 .0572 .0472 .0369	- 03339 - 03399 - 02265 - 02159 - 0152	.0672 .0657 .0552 .0441 .038		
2.9 4 1 2 2.9 8 1 6 3.0 2 6 1 3.0 7 5 9 3.1 3 2 4	2,0518 2,0918 2,1360 2,1854 2,2416	.990 .991 .992 .993 .994	259 236 2213 2189 2165	- 0083 - 0075 - 0068 - 0060 - 0051	.020 .019 .017 .015		
3.1 980 3.2 7 6 6 3.3 7 5 2 3.5 0 9 7 3.7 2 8 8	23068 23850 24833 26174 28362	995 996 997 998 999	0140 0115 0088 0061	- 0 0 4 3 - 0 0 3 5 - 0 0 2 6 - 0 0 1 8 - 0 0 0 9	.012 .010 .008 .005		
4.8386	39458	1.000	0000	0000	.000		

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

				<u>·</u>		
	$\beta = 2.0, S_{W} = -0.4$					
η	f	f'	f''	S	S!	
0 0381 0777 1190 1620	.0009 .0039 .0091	0 .050 .100 .150	1.3329 1.2868 1.2386 1.1884 1.1360	-0.4 0 0 0 - 3 9 1 2 - 3 8 2 0 - 3 7 2 5 - 3 6 2 6	0.230 .230 .230 .230 .230	
2071 2546 3049 3583 4155	.0268 .0399 .0563 .0764	.250 .300 .350 .400 .450	1.0816 1.0252 .9668 .9064 .8440	-3523 -3413 -3356 -3176 -3046	.230 .229 .229 .228 .227	
.4772 .5443 .6181 .7002 .7932	.1301 .1654 .2079 .2593	.500 .550 .600 .650	.7797 .7134 .6450 .5746	2906 2756 2592 2413 2215	.225 .223 .220 .216 .210	
.9011 1.0302 1.1921	.4005 .5007 .6345	.750 .800 .850	.4276 .3507 .2714	- 1992 - 1738 - 1442	.202 .191 .174	
1.4116 1.4671 1.5284 1.5970 1.6748	.8269 .8772 .9333 .9967	.900 .910 .920 .930 .940	.1890 .1721 .1549 .1376 .1200	-1087 -1007 -0922 -0833 -0739	.148 .141 .134 .125 .116	
1.7651 1.8164 1.8731 1.9365 2.0919 2.1914 2.3161	1.1 5 4 9 1.2 0 3 8 1.2 5 8 1 1.3 1 9 1 1.3 8 8 1.4 6 9 8 1.5 6 7 2 1.6 8 9 7	9965 99665 99766 99788 9988	.1 0 2 1 .0 9 3 0 .0 8 3 8 .0 7 4 4 .0 6 5 5 3 .0 4 5 4 .0 3 5 2	- 0640 - 0587 - 0533 - 0477 - 0419 - 0295 - 0229	.1048 .0985 .0857 .0688 .048	
2.4850 2.5276 2.5747 2.6274 2.6873	1.8 5 6 5 1.8 9 8 7 1.9 4 5 4 1.9 9 7 7 2.0 5 7 2	.990 .991 .992 .993	.0245 .0223 .0201 .0178	- 0158 - 0144 - 0129 - 0114 - 0099	.035 .032 .030 .027	
2.7 5 6 9 2.8 4 0 2 2.9 4 4 9 3.0 8 7 5 3.3 1 9 6	2.1 2 6 4 2.2 0 9 3 2.3 1 3 6 2.4 5 5 9 2.6 8 7 7	995 997 999 999	.0132 .0108 .0083 .0058	0083 0067 0051 0034 0017	.020 .017 .013 .009	
44979	3.8657	1.000	.0000	.0000	.000	

$\beta = -0.1947, S_{W} = 0$							
η	f	f'	f"				
0 .2 .4 .6 .8	0 0007 0019 0020 .0006	0 0061 0044 .0051 .0223	-0.0 5 0 0 0 1 1 1 .0 2 7 9 .0 6 6 9 .1 0 5 8				
1.0 1.2 1.4 1.6 1.8	.0075 .0201 .0400 .0688 .1077	.0474 .0801 .1205 .1681 .2226	.1 4 4 6 .1 8 2 9 .2 2 0 3 .2 5 5 9 .2 8 8 5				
2.0 2.2 2.4 2.6 2.8	.1582 .2214 .2981 .3889 .4942	.2833 .3490 .4186 .4903	.3169 .3395 .3547 .3610 .3574				
3.0 3.2 3.4 3.6 3.8	.6138 .7470 .8930 1.0506 1.2182	.6326 .6991 .7601 .8141 .8602	.3436 .3202 .2885 .2510 .2104				
4.0 4.2 4.4 4.6 4.8	1.39 4 1 1.57 6 9 1.7 6 5 0 1.9 5 7 0 2.1 5 1 8	.8983 .9284 .9512 .9679 .9796	.1697 .1316 .0980 .0700 .0480				
50 524 558 58	2.3 4 8 6 2.5 4 6 7 2.7 4 5 5 2.9 4 4 8 3.1 4 4 4	9875 9955 9999 9999 9999	.0315 .0199 .0121 .0070				
6.0 6.2 6.4 6.6 6.8	3.3 4 4 3 3.5 4 4 2 3.7 4 4 1 3.9 4 4 0 4.1 4 4 0	9999 9999 99998 9999	.0021				

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.1, S_{W} = 1.0$					
η	f	f'	f"	S	S'	
0 .2 .4 .6	0 0030 0108 0220 0349	0 0283 0498 0616 0670	-0.1613 1217 0831 0455 0086	1.0000 .9585 .9169 .8753 .8335	-0.2076 2076 2079 2086 2093	
1.0 1.2 1.4 1.6 1.8	0483 0605 0702 0759 0763	0651 0560 0397 0164 .0140	.0276 .0635 .0990 .1344 .1695	.7913 .7488 .7058 .6621	2115 2133 2167 2199 2233	
0 0 4 4 6 6 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	0699 0553 0312 .0037 .0506	.0514 .0956 .1464 .2034 .2658	.2042 .2376 .2699 .2992 .3247	.5728 .5272 .4611 .4347 .3884	2266 2295 2315 2322 2310	
3.0 3.2 3.4 3.6 3.8	.1104 .1839 .2718 .3742 .4909	.3329 .4033 .4756 .5481 .6188	.3448 .3582 .3634 .3595 .3461	.3 4 2 5 .2 9 7 6 .2 5 4 4 .2 1 3 5 .1 7 5 5	2273 2208 2110 1978 1815	
4.0 4.2 4.4 4.6 4.8	.6215 .7649 .9201 1.0855 1.2597	.6859 .7477 .8028 .8502 .8896	.3236 .2931 .2568 .2172 .1771	.1 4 1 0 .1 1 0 6 .0 8 4 5 .0 6 2 8 .0 4 5 3	1624 1414 1196 0979 0774	
5.0 5.2 5.4 5.6 5.8	1.4 4 0 9 1.6 2 7 7 1.8 1 8 6 2.0 1 2 7 2.2 0 8 9	.9211 .9455 .9635 .9763 .9850	.1390 .1048 .0760 .0529	.0317 .0215 .0141 .0090	0591 0435 0308 0210 0137	
6.0 6.2 6.4 6.6 6.8	2.4065 2.6050 2.8041 3.0036 3.2033	.9908 .9944 .9966 .9978 .9986	.0227 .0140 .0033 .0048	.0033 .0020 .0012 .0007	0087 0052 0031 0017 0009	
7.0 7.2 7.4 7.6 7.8	3.4 0 3 0 3.6 0 2 8 3.8 0 2 7 4.0 0 2 5 4.2 0 2 4	.9990 .9992 .9993 .9994	.0014 .0007 .0004 .0003	0003 0003 0003 0003	0005 0002 0001 0001	

	,	$\beta = -0.130$	05, S _w = 1.0		
η	f	f'	f"	S	S'
ુ જ 4	0 0007 0013 .0002 .0056	0 0048 .0004 .0155	-0.0500 .0014 .0511 .0992 .1456	1.0000 9372 .8744 .8117 .7489	-0.3139 3139 3139 3140 3138
1.0 1.2 1.4 1.6 1.8	.0168 .0356 .0637 .1025 .1536	.0736 .1159 .1661 .2237 .2876	.1900 .2319 .2704 .3046 .3331	.6862 .6237 .5616 .5004 .4404	3132 3116 3085 3085 3959
2.0 2.2 2.4 2.6 2.8	2179 2964 3896 4975 6199	.3565 .4268 .5026 .5762 .6472	.3545 .3672 .3702 .3627 .3449	.3822 .3266 .2741 .2256 .1817	2852 2709 2530 2316 2072
3.0 3.2 3.4 3.6 3.8	;7560 .9049 1.0650 1.2349 1.4128	.7135 .7737 .8263 .8707	.3176 .2828 .2430 .3012 .1603	.1 4 2 8 .1 0 9 5 .0 8 1 6 .0 5 9 1 .0 4 1 4	1806 1530 1257 0999 0767
4.0 4.2 4.4 4.6 4.8	1.5971 1.7863 1.9792 2.1746 2.3718	.9351 .9563 .9716 .9822 .9892	.1227 .0903 .0637 .0432 .0281	.0 2 8 2 .0 1 8 6 .0 1 1 8 .0 0 7 2 .0 0 4 3	0 5 6 7 0 4 0 4 0 2 7 7 0 1 8 3 0 1 1 6
5.0 5.2 5.4 5.6 5.8	2.5701 2.7692 2.9686 3.1684 3.3682	.9937 .9964 .9981 .9990	0175 0105 0061 0034 0018	.0025 .0014 .0007 .0004 .0002	0 0 7 1 0 0 4 2 0 0 2 4 0 0 1 3 0 0 0 7
6.0 6.2 6.4	3.5681 3.7681 3.9681	.9997 .9999 .9999	.0009 .0005 .0002	.0001	0003 0001 0001

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = -0.1295, S_{W} = 1.0$						
η	f	f'	f"	S			
0 .2 .4 .6 .8	0 .0003 .0027 .0091	0 .0051 .0203 .0450	.0509 .1001 .1473 .1924	1.0000 .9322 .8645 .7967 .7291	-0.3389 3389 3388 3384 3374		
1.0 1.2 1.4 1.6 1.8	.0413 .0706 .1108 .1634 .2293	.1218 .1727 .2309 .2953 .3647	.2347 .2735 .3076 .3358 .3565	.6618 .5951 .5293 .4650 .4027	3354 3317 3258 3170 3049		
2.0 2.4 2.6 2.8	.3094 .4043 .5139 .6380 .7757	.4373 .5114 .5847 .6553 .7211	.3684 .3702 .3615 .3425 .3141	.3 4 3 3 .2 8 7 4 .2 3 5 9 .1 8 9 4 .1 4 8 5	2890 2691 2456 2189 1901		
3.0 3.2 3.4 3.6 3.8	9260 1.0874 1.2583 1.4371 1.6221	.7804 .8321 .8756 .9107 .9381	.2784 .2382 .1963 .1556 .1185	.1 1 3 4 .0 8 4 3 .0 6 0 8 .0 4 2 6 .0 2 8 9	1604 1312 1038 0793 0584		
4.0 4.2 4.4 4.6 4.8	1.8 1 1 8 2.0 0 5 1 2.2 0 0 8 2.3 9 8 1 2.5 9 6 6	.9585 .9731 .9832 .9899 .9941	.0867 .0609 .0410 .0265 .0165	.0189 .0120 .0074 .0044 .0026	0 414 0 282 0 185 0 117 0 0 71		
5.0 5.2 5.4 5.6 5.8	2.7957 2.9952 3.1949 3.3948 3.5947	.9967 .9982 .9991 .9995	.0098 .0056 .0031 .0017	.0015 .0009 .0005 .0003	0 0 4 2 0 0 2 3 0 0 1 3 0 0 0 7 0 0 0 3		
6.0 6.2 6.4	3,7947 3,9947 4,1947	.9999 1.0000 1.0000	.0004 .0001 .0003	.0002 .0002 .0001	0001 0001 0001		

$\beta = -0.1, S_{W} = 1.0$						
η	f	f'	f''	S	S'	
0 .2451 .4477 .6260	0 .0057 .0207 .0429 .0712	0 .050 .100 .150 .200	0.1805 .2284 .2654 .2954 .3199	1.0000 .9012 .8197 .7482 .6836	-0.4 0 3 3 4 0 2 7 4 0 1 6 3 9 9 3 3 9 5 6	
.9400 1.0837 1.2220 1.3568 1.4896	.1053 .1448 .1897 .2402 .2967	.250 .300 .350 .400	.3398 .3554 .3670 .3745 .3778	.6241 .5684 .5160 .4662 .4186	3904 3834 3745 3640	
1.6220 1.7555 1.8918 2.0329 2.1815	.3595 .4296 .5080 .5962 .6966	.5 0 0 .5 5 0 .6 0 0 .6 5 0 .7 0 0	.3769 .3716 .3615 .3463 .3255	.3731 .3293 .2872 .2465 .2071	336 319 299 277 251	
2.3416 2.5192 2.7250	.8128 .9505 1.1204	.750 .800 .850	.2985 .2642 .2215	.1691 .1324 .0969	223; 190; 154;	
29818 30436 31105 31839 32656	1.3 4 5 5 1.4 0 1 3 1.4 6 2 5 1.5 3 0 4 1.6 0 6 9	900 910 920 930 940	.1682 .1560 .1431 .1295 .1152	.0628 .0561 .0496 .0430 .0366	112 103 093 084 073	
3,3586 3,4107 3,4676 3,5305 3,6011 3,6821 3,7779 3,8964	1.6948 1.7444 1.7989 1.8594 1.9278 2.0066 2.1002 2.2166	99999999 999999999	.1000 .0920 .0838 .0753 .0664 .0571 .0474	0302 0270 0239 0208 0177 0146 0116	063 058 058 046 040 034 028	
4.0551 4.0950 4.1388 4.1878 4.2433	2,3734 2,4128 2,4563 2,5049 2,5601	99999 99999 9999	.0262 .0239 .0216 .0192	0056 .0050 .0045 .0039	015 014 012 011 009	
4.3077 4.3847 4.4813 4.6129 4.8269	2.6 2 4 1 2.7 0 0 8 2.7 9 7 1 2.9 2 8 3 3.1 4 2 0	567 999999 99999	.0143 .0117 .0090 .0062	.0027 .0022 .0016 .0010	008 006 005 003 001	
5.9066	4,2215	1,000	.0000	.0000	.000	

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

[β = 0.3,	S _w = 1.0		
η	f	f'	f"	S	S¹
0 1 2 3 .4	0 .0048 .0189 .0416	0 .0953 .1848 .2685 .3466	0.9829 .9237 .8657 .8089 .7531	1.0000 .9454 .8909 .8365 .7823	-0.5 4 5 7 5 4 5 6 5 4 5 0 5 4 3 4 5 4 0 3
.5 .6 .7 .8	.1107 .1560 .2078 .2655 .3286	.4192 .4863 .5481 .6047 .6563	.6983 .6445 .5919 .5406 .4909	.7285 .6752 .6229 .5716 .5216	5354 5284 5189 5067 4919
1.0 1.1 1.2 1.3 1.4	.3966 .4690 .5454 .6254	.7029 .7449 .7625 .8157 .8450	.4 4 3 0 .3 9 7 1 .3 5 3 5 .3 1 2 4 .2 7 4 0	.4733 .4268 .3825 .3405 .3011	4744 4544 4319 4074 3811
1.5 1.6 1.7 1.8 1.9	.7942 .8824 .9727 1.0647 1.1582	.8706 .8928 .9119 .9281 .9419	.2384 .2058 .1761 .1494 .1256	.2643 .2304 .1993 .1711 .1458	3535 3251 2963 2676 2395
2.0 2.1 2.2 2.3 2.4	1.2530 1.3489 1.4456 1.5430 1.6409	.9534 .9629 .9708 .9771 .9823	.1 0 4 8 .0 8 6 5 .0 7 0 8 .0 5 7 4 .0 4 6 0	.1232 .1033 .0859 .0708	2123 1864 1621 1396 1190
2.5 2.6 2.7 2.8 2.9	1.7394 1.8382 1.9373 2.0366 2.1361	.9864 .9897 .9922 .9942 .9957	0366 0288 0224 0173 0132	.0470 .0377 .0301 .0238	1005 0841 0696 0571 0463
3.0 3.1 3.2 3.3 3.4	2.2 3 5 8 2.3 3 5 5 2.4 3 5 3 2.5 3 5 2 2.6 3 5 1	.9969 .9977 .9984 .9989	.0100 .0075 .0055 .0040	.0 1 4 4 .0 1 1 1 .0 0 8 5 .0 0 6 4 .0 0 4 8	0372 0296 0233 0182 0141
3.5 3.6 3.7 3.8 3.9	2.7 3 5 0 2.8 3 5 0 2.9 3 4 9 3.0 3 4 9 3.1 3 4 9	.9995 .9996 .9998 .9998	.0021 .0015 .0010 .0007	.0036 .0026 .0019 .0014 .0010	0107 0081 0061 0045 0033
4.0 4.1 4.2 4.3 4.4	3.2349 3.3349 3.4349 3.5349 3.6349	.9999 1.0000 1.0000 1.0000	.0003 .0002 .0001 .0001	.0007 .0005 .0004 .0003	0024 0017 0012 0009 0006
4.5 4.6 4.7	3.7349 3.8349 3.9349	1.0000 1.0000 1.0000	.0000 .0000	.0001 .0001 .0001	0004 0003 0002

	$\beta = 0.5, S_W = 1.0$					
η	f	f'	f"	S	S'	
0 2 4 4 8	0 .0234 .0885 .1879 .3151	.2274 .4172 .5720 .6950	1.2 3 5 1 1.0 4 0 9 .8 5 9 1 .6 9 1 8 .5 4 1 4	1.0000 .8855 .7717 .6599 .5523	-0.5725 5716 5656 5505 5237	
1.0 1.2 1.4 1.6 1.8	.4640 .6295 .8069 .9929 1.1846	.7898 .8605 .9113 .9462 .9692	.4103 .3000 .2108 .1417 .0908	.4513 .3592 .2780 .2088 .1521	4846 4346 3767 3147 2532	
2.0 2.2 2.4 2.6 2.8	1.3800 1.5776 1.7765 1.9761 2.1760	.9835 .9920 .9966 .9990	.0550 .0313 .0165 .0078	.1072 .0732 .0484 .0310 .0193	1 9 5 9 1 4 5 7 1 0 4 1 0 7 1 4 0 4 7 1	
3.0 3.2 3.4 3.6 3.8	2.3761 2.5762 2.7762 2.9763 3.1764	1.0004 1.0004 1.0004 1.0003	.0009 0001 0004 0004	.0117 .0069 .0041 .0025	0299 0182 0106 0061 0032	
4.0 4.2 4.4 4.6 4.8	3.3764 3.5764 3.7765 3.9765 4.1765	1.0002 1.0001 1.0001 1.0001	0002 0002 0001 0001	.0011 .0008 .0007 .0006	0017 0008 0004 0002 0001	
5.0 5.2 5.4 5.6	4.3765 4.5765 4.7765 4.9765	1.0001 1.0000 1.0000 1.0000	0001 0001 0001 0001	.0006 .0006 .0006	.0000 0000 0000 0000	

TABLE 1. - Continued. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

	$\beta = 1.0, S_W = 1.0$					
η	f	f'	f"	S	Si	
0 .1 .2 .3 .4	.0 0 8 4 .0 3 2 1 .0 6 9 4 .1 1 8 5	0 .1638 .3084 .4347 .5439	1.7368 1.5403 1.3526 1.1758 1.0114	1.0 0 0 0 .9 3 8 4 .8 7 7 0 .8 1 5 7 .7 5 4 9	-0.6154 6152 6140 6110 6053	
5.67.89	.1776 2455 3205 .4016 .4876	.6374 .7165 .7825 .8370 .8812	.8603 .7232 .6004 .4915 .3965	.6948 .6357 .5781 .5223 .4687	5965 5840 5677 5476 5238	
1.0 1.1 1.2 1.3 1.4	.5776 .6707 .7663 .8637 .9625	.9167 .9445 .9659 .9820 .936	.3142 .2442 .1857 .1372 .0979	.4176 .3694 .3243 .2826 .2444	4967 4666 4343 4003 3654	
1.5 1.6 1.7 1.8 1.9	1.0 6 2 3 1.1 6 2 8 1.2 6 3 7 1.3 6 4 8 1.4 6 6 1	1.0019 1.0073 1.0105 1.0122 1.0126	.0666 .0422 .0237 .0101 .0004	.2096 .1783 .1505 .1259 .1045	3302 2954 2617 3294 1992	
2.0 2.1 2.2 2.3 2.4	1.5673 1.6685 1.7696 1.8706 1.9714	1.0123 1.0115 1.0104 1.0091 1.0078	0061 0101 0123 0131 0130	.0860 .0702 .0568 .0456	1711 1456 1226 1022 0843	
5.6.7.8.9. 0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.	2.0722 2.1728 2.2732 2.3736 2.4739	1.0065 1.0053 1.0043 1.0034 1.0026	0122 0111 0098 0085 0071	.0286 .0224 .0174 .0134 .0103	0689 0557 04464 0277	
3.0 3.1 3.2 3.3 3.4	2.5741 2.6743 2.7744 2.8745 2.9746	1.0020 1.0014 1.0010 1.0006 1.0003	0059 0049 0040 0032 0026	.0078 .0059 .0045 .0034	0216 0166 0126 0095 0071	
3.5 3.6	3.0745 3.1746	1.0001	0020 0016	.0019 .0015	0053 0038	

$\beta = 1.5, S_W = 1.0$						
η	f	f'	f"	S	S¹	
0 .1 .2 .3 .4	0 .0102 .0389 .0633 .1409	.1992 .3699 .5139	2.1 4 0 2 1.8 4 6 4 1.5 6 9 7 1.3 1 5 0 1.0 8 5 4	1.0000 .9358 .8716 .3077 .7443	-0.6425 0403 0400 0370 5300	
.5 .6 .7 .8	2093 2866 3709 4608 5550	.7318 .8109 .8736 .9221 .9590	.8818 .7044 .5521 .4234 .3163	.6518 .6205 .5611 .5039 .4491	6191 6040 5845 5607 5329	
1.0 1.1 1.2 1.3 1.4	.6523 .7519 .8531 .9554 1.0583	.9861 1.0053 1.0182 1.0262 1.0305	.2287 .1582 .1026 .0598	.3974 .3489 .3039 .2626 .2851	5 0 1 7 4 6 7 7 4 3 1 6 3 9 4 3 3 5 0 6	
1.5 1.6 1.7 1.8 1.9	1.1614 1.2646 1.3677 1.4706 1.5731	1.0 3 2 1 1.0 3 1 6 1.0 2 9 9 1.0 2 7 4 1.0 2 4 4	.0044 0117 0221 0282 0310	,1913 ,1612 ,1347 ,1116	3191 2827 2478 2150 1647	
2.0 2.1 2.2 2.3 2.4	1.6754 1.7774 1.8791 1.9805 2.0816	1.0 2 1 2 1.0 1 5 1 1.0 1 5 2 1.0 1 2 6 1.0 1 0 2	0314 0302 0279 0250 0219	.0746 .0601 .0480 .0380	1570 1321 1100 0907 0740	
2.5 2.6 2.7 2.8 2.9	2.1 8 2 5 2.2 8 3 2 2.3 8 3 8 2.4 8 4 3 2.5 8 4 6	1.0082 1.0065 1.0050 1.0039 1.0029	0 1 8 8 0 1 5 8 0 1 3 0 0 1 0 6 0 0 8 5	.0231 .0178 .0135 .0101 .0075	0598 0478 0379 0297 0231	
3.0 3.1 3.2 3.3 3.4	2.6 8 4 9 2.7 8 5 0 2.8 6 5 2 2.9 8 5 3 3.0 6 5 3	1.0022 1.0016 1.0011 1.0007 1.0005	0067 0052 0040 0031 0023	.0055 .0039 .0028 .0019	0177 0135 0101 0076 0056	
3.5 3.6 3.7 3.8	3.1 8 5 4 3.2 8 5 4 3.3 8 5 4 3.4 8 5 4	1.0003 1.0001 1.0000	0017 0013 0009 0007	.0008 .0004 .0002	0041 0030 0021 0015	

TABLE 1. - Concluded. SIMILAR SOLUTIONS OF LAMINAR COMPRESSIBLE BOUNDARY-LAYER EQUATIONS.

		$\beta = 2.0,$	S _w = 1.0		·
η	f	f'	f"	S	S'
0 .1 .2 .3 .4	0. .0118 .0446 .0947 .1589	.2291 .4204 .5771 .7031	2.4878 2.0971 1.7341 1.4072 1.1204	1.0000 .9339 .8678 .8021 .7370	-0.6613 6611 6593 6548 6466
.5 .6 .7 .8	.2344 .3187 .4097 .5056	.8025 .8793 .9372 .9795 1.0094	.8743 .6671 .4958 .3566 .2455	.6729 .6103 .5497 .4915 .4362	6341 6168 5948 5682 5375
1.0 1.1 1.2 1.3 1.4	.7072 8108 .9153 1.0203 1.1254	1.0 2 9 4 1.0 4 1 8 1.0 4 8 4 1.0 5 0 8 1.0 5 0 2	.1587 .0921 .0425 .0067	.3841 .3356 .2909 .2500 .2132	5034 4666 4280 3885 3490
1.5 1.6 1.7 1.8 1.9	1,2303 1,3349 1,4390 1,5426 1,6458	1.0 4 7 5 1.0 4 3 6 1.0 3 9 0 1.0 3 4 1 1.0 2 9 3	0 3 4 1 0 4 3 5 0 4 7 8 0 4 8 6 0 4 6 8	.1802 .1511 .1256 .1035 .0846	3102 2729 2375 2046 1745
2.0 2.1 2.2 2.3 2.4	1.7485 1.8508 1.9527 2.0542 2.1554	1.0248 1.0207 1.0170 1.0138 1.0111	0 4 3. 4 0 3 9 1 0 3 4 4 0 2 9 6 0 2 5 0	.0685 .0550 .0438 .0346 .0271	1 4 7 3 1 2 3 0 1 0 1 7 0 8 3 2 0 6 7 4
2.5 2.6 2.7 2.8 2.9	2.2 5 6 4 2.3 5 7 2 2.4 5 7 8 2.5 5 8 3 2.6 5 8 7	1.0088 1.0069 1.0054 1.0041 1.0032	0208 0170 0137 0109 0086	.0210 .0162 .0123 .0094 .0070	0541 0429 0338 0263 0202
3.0 3.1 3.2 3.3 3.4	2.7589 2.8591 2.9593 3.0594 3.1595	1.0024 1.0018 1.0014 1.0011 1.0008	0066 0051 0038 0028 0021	.0053 .0039 .0029 .0022	0154 0117 0087 0064 0047
3.5 3.6 3.7 3.8 3.9	3.2596 3.3596 3.4597 3.5597 3.6598	1.0006 1.0005 1.0004 1.0004 1.0003	0015 0011 0007 0005 0003	.0012 .0009 .0007 .0006	0034 0025 0018 0012 0009
4.0	3.7598	1.0003	0002	.0004	0006

TABLE II. - SUMMARY OF HEAT-TRANSFER

AND WALL-SHEAR PARAMETERS.

Sw	β	f"	S,	Crew Nu
-1.0	-0.326 3657 3884 360 30 14 0 .50 2.00	0 .050 .140 .2448 .3182 .4166 .470 .5806 .7381	0.2477 .2958 .3527 .4001 .4262 .4554 .470 .4948 .5203	0 .3381 .7939 1.224 1.493 1.830 2.000 2.347 2.837
-0.8	-0.10 2685 3088 325 3285 325 30 14 0 .50 1.50 2.00	-0.0686 050 0 .0493 .0693 .110 .1354 .2086 .3841 .470 .6547 .8689 .9480	0.0447 .1829 .2261 .2545 .2644 .2818 .2913 .3155 .359 .376 .403 .4261 .4331	-2.456 4374 0 .3100 .4194 .6245 .7438 1.058 1.712 2.000 2.599 3.263 3.502
-0.4	-0.235 246 2483 24 20 0 .50 2.00	-0.050 0 .050 .1064 .2183 .470 .7947 1.3329	0.0447 .1249 .1360 .1474 .1626 .188 .209 .2304	-0.8949 0 .2941 .5775 1.074 2.000 3.042 4.628
0	-0.1947 1988 16 0 .50 1.00 1.60 2.00	-0.050 0 .1905 .470 .9277 1.2326 1.5213 1.6870	0 0 0 0 0 0 0 0 0	a 0 .9480 2.000 3.436 4.317 5.122 5.565
1.0	-0.10 1305 1295 10 0 .30 .50 1.00 1.50 2.00	-0.1613 050 0 .1805 .470 .9829 1.2351 1.7368 2.1402 2.4878	3139 3388 4033 470 5457 5725 6154 6425	-1.554 3186 0 .8956 2.000 3.602 4.315 5.644 6.662 7.527

^a This value was not calculated.

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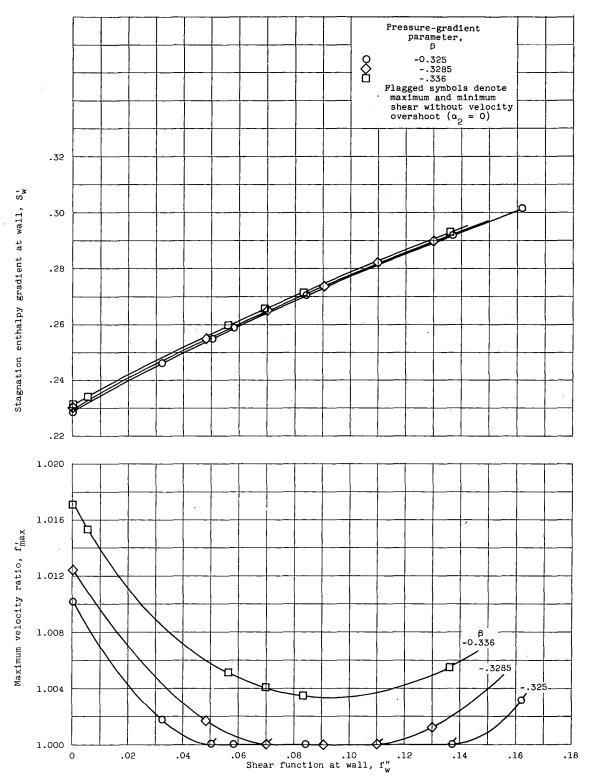
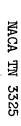


Figure 1. - Family of solutions for adverse pressure gradient. $S_{\rm W}$ = -0.8.



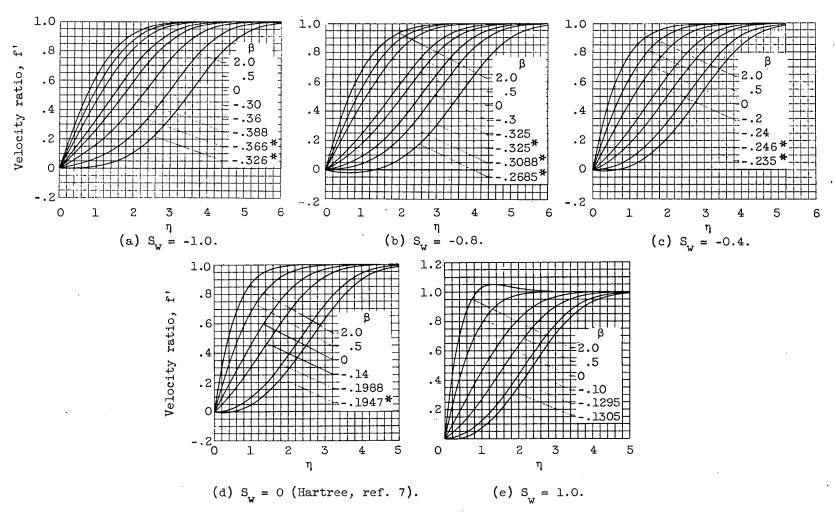


Figure 2. - Velocity profiles as function of similarity variable η. (* denotes lower-branch solutions.)

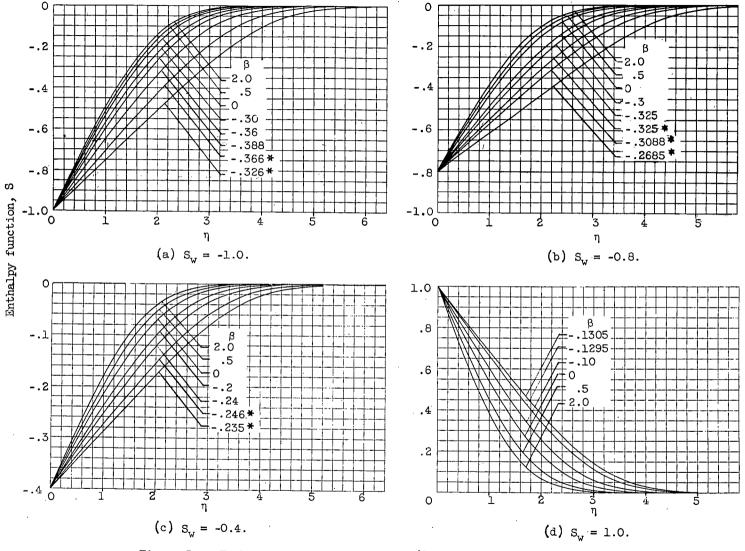


Figure 3. - Enthalpy function profiles. (* denotes lower-branch solutions.)

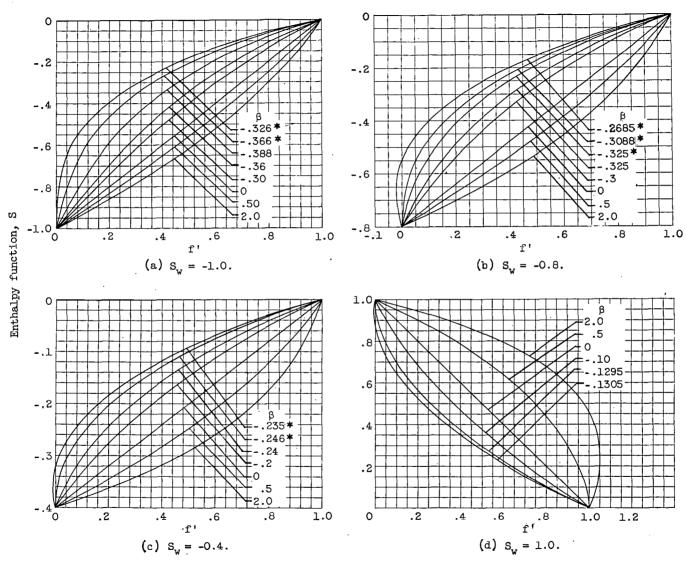


Figure 4. - Enthalpy function representation in velocity plane. (* denotes lower-branch solutions.)

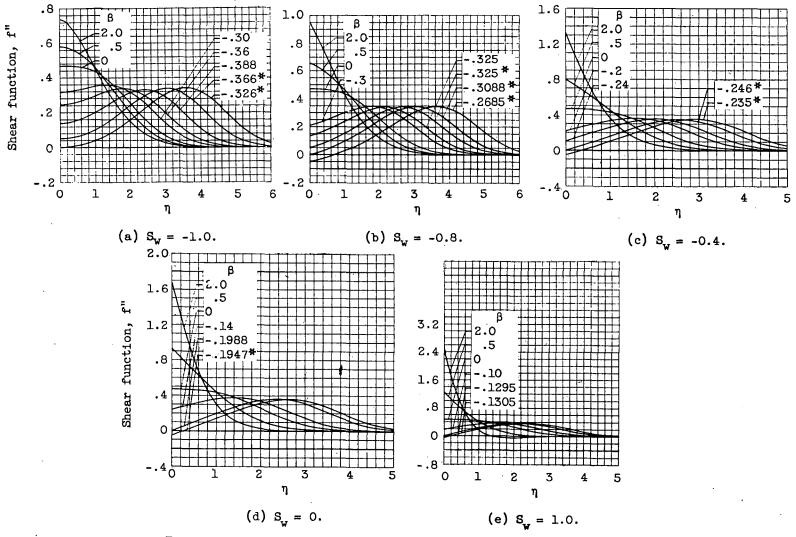
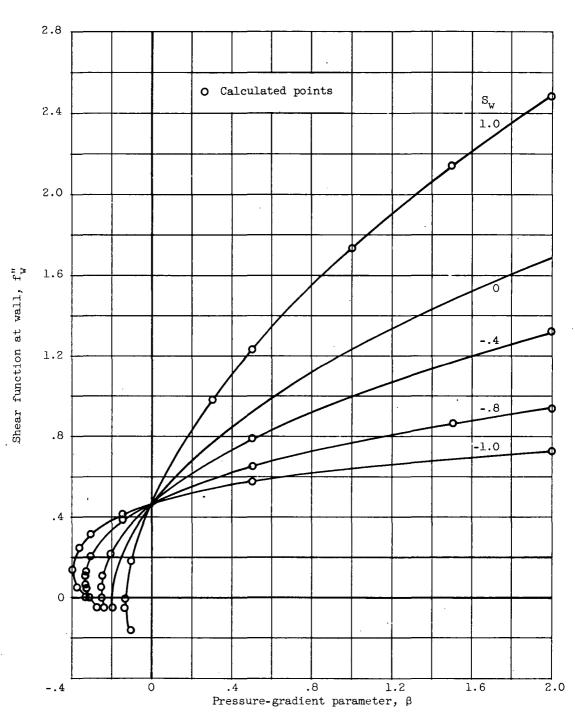
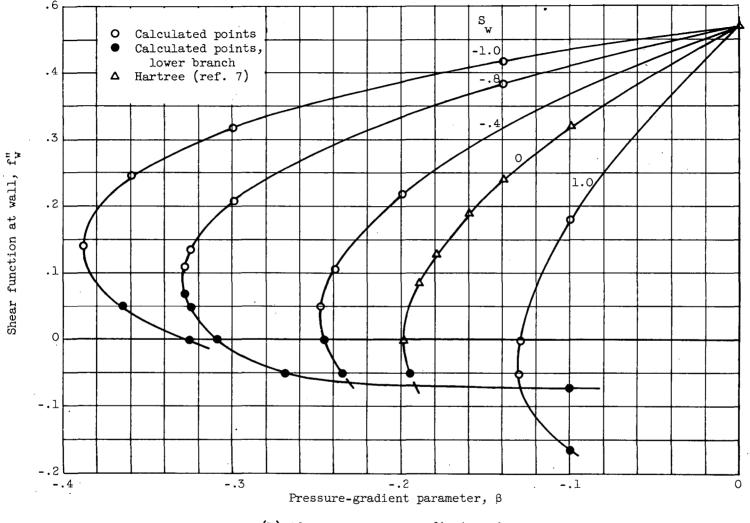


Figure 5. - Shear-function profiles. (* denotes lower-branch solutions.)



(a) Favorable and adverse pressure gradients.

Figure 6. - Effect of pressure gradient on wall shear.



(b) Adverse pressure-gradient region.

Figure 6. - Concluded. Effect of pressure gradient on wall shear.

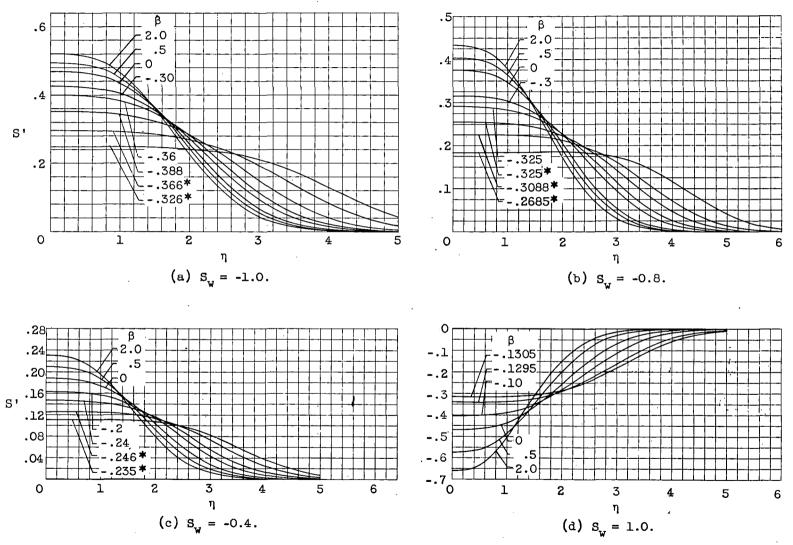


Figure 7. - Stagnation enthalpy gradient across boundary layer. (* denotes lower-branch solutions.)

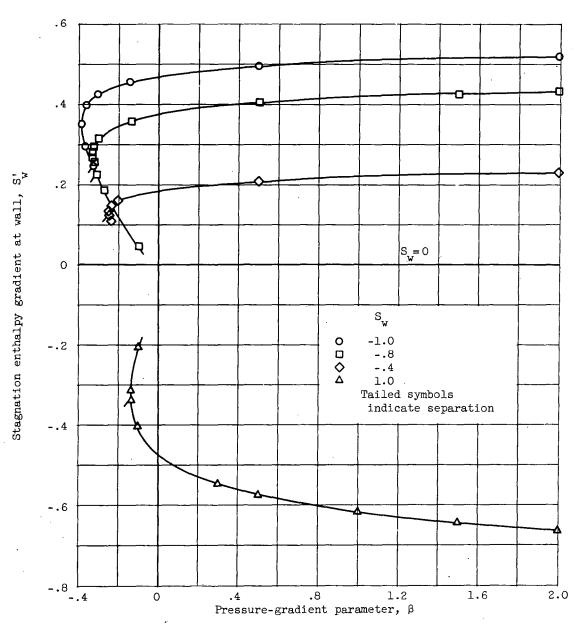


Figure 8. - Variation of heat transfer with pressure gradient.

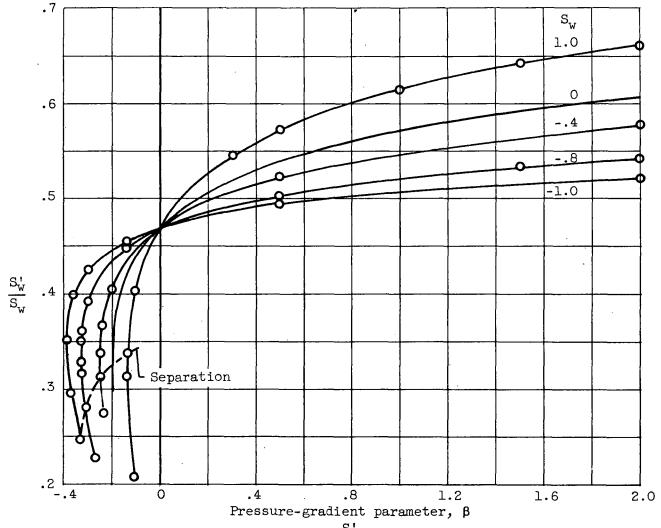


Figure 9. - Variation of $\frac{S_{\underline{w}}'}{S_{\underline{w}}}$ with pressure gradient.

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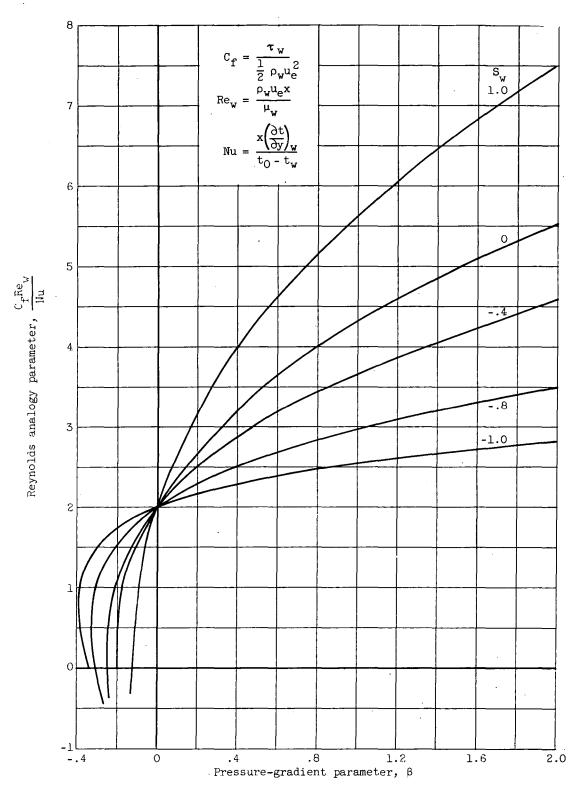


Figure 10. - Variation of Reynolds analogy parameter with pressure gradient.