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# TECHNICAL NOTE

D-111

FATIGUE STRENGTHS OF AIRCRAFT MATERIALS

AXIAL-LOAD FATIGUE TESTS ON EDGE-NOTCHED SHEET

SPECIMENS OF 2024-T3 AND 7075-T6 ALUMINUM

ALLOYS AND OF SAE 4130 STEEL WITH NOTCH

RADI OF 0.004 AND 0.070 INCH

By H. J. Grover, W. S. Hyler, and L. R. Jackson

Battelle Memorial Institute

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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## SUMMARY

The present report gives results of axial-load fatigue tests on notched specimens of three sheet materials: 2024-T3 and 7075-T6 aluminum alloys and normalized SAE 4130 steel.

Two edge-notched specimens were designed and tested, each having a theoretical stress-concentration factor  $K_t = 4.0$ . The radii of the notches were 0.004 and 0.070 inch. Tests of these specimens were run at two levels of nominal mean stress: 0 and 20,000 psi.

Results of these studies extended information previously reported on tests of specimens with varying notch severity. They afford data on the variation of fatigue-strength reduction with notch radius and on the potential usefulness of Neuber's technical stress-concentration factor  $K_n$ .

## INTRODUCTION

During the approximate period 1947 to 1952 a broad program was carried out at Battelle Memorial Institute under the sponsorship and with the financial assistance of the National Advisory Committee for Aeronautics and at the Langley Aeronautical Laboratory of the NACA that involved a study of the fatigue strengths of metals commonly used in aircraft construction. A major objective of the investigation was to obtain basic data on the fatigue strengths of three sheet materials: 2024-T3 and 7075-T6 aluminum alloys, and normalized SAE 4130 steel.

From the Battelle studies five previous reports (refs. 1 to 5) present the following data:

- (1) Results of fatigue tests on unnotched sheet specimens (ref. 1)
- (2) Results of fatigue tests on sheet specimens notched with three different types of notches, including edge notches, having  $K_t = 2.0$ , and on sheet specimens notched with two different types of notches, including edge notches, having  $K_t = 4.0$  (ref. 2)
- (3) Results of fatigue tests on sheet specimens with severe edge notches having  $K_t = 5.0$  (ref. 3)
- (4) Results of fatigue tests on sheet specimens with mild edge notches having  $K_t = 1.5$  (ref. 4)
- (5) Results of fatigue tests on unnotched specimens of 2024-T3 and 7076-T6 aluminum alloys as determined in several laboratories (ref. 5)

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During the last year of the program a study was made of the effect of notch root radius on the fatigue behavior of the three materials. The specimens contained edge notches with  $K_t = 4.0$  (specimens with two edge notches having different root radii were tested). The data from this study have not been previously published; therefore, to provide complete documentation of the broad program, the present report has been prepared.

The authors wish to thank Mr. Paul Kuhn, then a member of the Structures Research Division of the NACA Langley Aeronautical Laboratory, for his help and guidance during this investigation.

#### SYMBOLS

A	Neuber parameter, dimension of "building blocks" of material, in.
$K_f$	fatigue notch factor, ratio of maximum stress for unnotched specimen to nominal maximum stress for notched specimen at same load ratio R and lifetime
$K_n$	Neuber technical stress-concentration factor
$K_t$	theoretical stress-concentration factor

R	load ratio, ratio of minimum stress to maximum stress in cycle
r	notch radius

#### EXPERIMENTAL PROCEDURE

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The experimental procedure in the work described in this report was generally the same as that in the previous investigations of unnotched and notched specimens (refs. 1 to 4). In this program fatigue tests on the notched specimens were run at only two levels of mean stress: 0 and 20,000 psi. The materials used were supplied from selected stock retained for this purpose at the Langley Aeronautical Laboratory of the NACA. Coupons were cut from 0.090-inch-thick commercial sheets of 2024-T3 and 7075-T6 aluminum alloys and from 0.075-inch-thick commercial sheets of normalized SAE 4130 steel. Static-strength properties of notched and unnotched sheet are given in table 1.

Figure 1 is a dimensional drawing of the notched specimens used in the fatigue tests. The dimensions of the notch were chosen, on the basis of available information, to give  $K_t = 4.0$ . Each notch was cut with tools especially designed to produce the contour desired. Machining cuts were successively lighter, so that the depth of the last two cuts was about 0.0005 inch. The notched specimens were electropolished after machining; 0.0008 inch of material was removed by this technique. All specimens were examined by a microscope comparator after electropolishing; the dimensions shown in figure 1 are representative of those measured during such examination. Figures 2 and 3 are photographs of typical fractures obtained with these specimens.

Fatigue tests were run on Krouse direct repeated-stress testing machines at speeds in the range 1,100 to 1,500 cycles per minute. A description of these machines is given in reference 1. The estimated precision of load measurement and maintenance was about  $\pm 3$  percent in tension-tension tests. In tests involving reversal of load, sheet specimens were restrained from buckling by the use of guide plates. The estimation of precision of loading was indirect in these cases; it is believed that error in load value in reversed-load testing did not exceed  $\pm 5$  percent.

#### RESULTS OF FATIGUE TESTS

Results of axial-load fatigue tests on edge-notched specimens with  $K_t = 4.0$  and  $r = 0.004$  and 0.070 inch are given in table 2. The

results are plotted in the form of S-N curves in figure 4. All stress values reported are nominal net-area stresses.

#### DISCUSSION

The results of the study shown in figure 4 have been summarized in tabular form (see table 3). Data from reference 2 for specimens with an edge notch with  $K_t = 4.0$  and  $r = 0.057$  inch are included in the table. There is evidence from the tabulated data that for a particular lifetime at a specified mean stress the fatigue strength decreases as the notch radius increases. The magnitude of the decrease differs from material to material.

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The fatigue-strength reduction factors for various lifetimes and mean stresses have been calculated for the above data and are listed in table 4. In the calculations, the following definition for  $K_f$  has been used:

$$K_f = \frac{\text{Maximum stress for unnotched specimen}}{\text{Nominal maximum stress for notched specimens at same load ratio } R \text{ and lifetime}}$$

(This definition for  $K_f$  has been used in refs. 2, 3, and 4. Load ratio  $R$  is the ratio of the minimum stress to the maximum stress of the stress cycle.) Stress values for unnotched specimens of the two aluminum alloys tested at  $R = -1.0$  were taken from table IV of reference 6. The remaining unnotched-specimen data came from references 1 and 5. The values in table 4 indicate the following trends:

(1) For a specified nominal mean stress,  $K_f$  is usually greatest at the longest lifetime and decreases with decreasing lifetime or increasing maximum stress.

(2) For a specified lifetime,  $K_f$  does not seem to vary in a consistent manner with mean stress.

(3) In comparing  $K_f$  values of notches with  $r = 0.004$  inch with those of notches with either  $r = 0.057$  or  $0.070$  inch it appears that, for a specified lifetime and nominal mean stress,  $K_f$  generally increases with increasing notch radius. This trend is somewhat less certain for the aluminum alloys when  $K_f$  values of notches having  $r = 0.057$  and  $0.070$  inch are compared.

The trend noted in item (3) would be expected on the basis of the Neuber expression concerning the effect of notch radius on the ideal stress-concentration factor (ref. 7). Neuber ascribes this effect to the breakdown of the theory of elasticity as notch radius approaches elementary crystal dimensions. The large changes in stress over extremely short distances in this case make more noticeable the effects of anisotropy and inhomogeneity. Neuber's treatment presupposes that, in the limiting condition where notch radius is zero (such as a V-notch), a material is composed of "building blocks" of finite dimension A, which he considered a constant for each material. Using this concept, he developed the following expression for a technical or "practical" stress-concentration factor  $K_n$  for notches with zero flank angle in terms of the ideal factor  $K_t$ , notch radius  $r$ , and parameter A:

$$K_n = 1 + \frac{K_t - 1}{1 + \sqrt{\frac{A}{r}}}$$

Examination of this function shows that for constant values of  $K_t$ ,  $K_n$  will decrease as  $r$  decreases.

Landers and Hardrath have indicated in reference 6 that for hole-notched specimens of 2024-T3 and 7075-T6 aluminum alloys the parameter  $A = 0.02$  inch provides good agreement between  $K_n$  and  $K_f$ , the fatigue notch factor. Experimental values of  $K_f$  at  $10^7$  cycles and zero mean stress for 2024-T3 and 7075-T6 aluminum alloys and SAE 4130 steel are plotted in figure 5. Curves of the above expression for  $K_n$ , with  $K_t = 4.0$  and  $A = 0.02$  inch, are also shown in the figure. The  $K_f$  values for the two aluminum alloys in figures 5(a) and 5(b) show good agreement with  $K_n$  obtained using  $A = 0.02$  inch. However,  $K_f$  values for steel are quite different from the  $K_n$  curve when  $A = 0.02$  inch is used. Adopting the approach for steel of Kuhn and Hardrath (ref. 8), a value of the Neuber parameter  $A = 0.0027$  inch was selected. The broken curve in figure 5(c) computed with  $A = 0.0027$  inch shows that there is a fit of the steel data with the Neuber concept comparable to that observed for the aluminum alloys.

To pursue this somewhat further,  $K_f$  values obtained during the broad program on edge-notched specimens for other values of  $K_t$  are plotted in figure 6 for the three materials along with the values obtained in the present investigation. The  $K_f$  values in the figures are not quite the same as those reported in references 1 to 4; instead, they have

been recomputed in accordance with the revised unnotched-specimen data of reference 5. The solid lines in each figure represent  $K_n$  as a function of root radius  $r$  for the four notch severities studied. For the two aluminum alloys (figs. 6(a) and 6(b)), the Neuber parameter was  $A = 0.02$  inch; whereas, for steel,  $A = 0.0027$  inch.

From these figures it would appear that  $K_n$  can indicate the approximate trend of  $K_f$  for the range of notch severities and root radii investigated for the three materials. For each of the materials it appears that  $K_n$  is somewhat greater than  $K_f$  for small values of  $K_t$ . At high values of  $K_t$ ,  $K_n$  is somewhat smaller than  $K_f$ . Thus,  $K_n$  would provide a conservative estimate of  $K_f$  for small values of  $K_t$  and an unconservative estimate for high values of  $K_t$ .

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### CONCLUSIONS

The following conclusions appear warranted from this study of axial-load fatigue tests on notched sheet specimens of 2024-T3 and 7075-T6 aluminum alloy and SAE 4130 steel:

1. For a constant value of theoretical stress-concentration factor  $K_t$ , namely,  $K_t = 4.0$ , the fatigue notch factor  $K_f$  generally increases as the root radius increases.

2. These fatigue notch factors for specimens with edge notches with  $K_t = 4.0$  are in reasonable agreement with  $K_n$  values for notches with zero flank angle computed from

$$K_n = 1 + \frac{K_t - 1}{1 + \sqrt{\frac{A}{r}}}$$

where  $r$  is the notch radius and the parameter  $A = 0.02$  inch for the two aluminum alloys and  $0.0027$  inch for normalized SAE 4130 steel.

3. In considering all data assembled during the broad program on specimens with edge-type notches with various values of  $K_t$  and root radius, the Neuber factor  $K_n$  was shown to be helpful in predicting the trend of the data using the above values of  $A$ . There was evidence,



however, that at high values of  $K_t$  the predictions would be unconservative, whereas, at low values of  $K_t$ , the predictions would be conservative.

Battelle Memorial Institute,  
Columbus, Ohio, August 8, 1958.

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7. Neuber, Heinz: Theory of Notch Stresses: Principles for Exact Stress Calculation. J. W. Edwards (Ann Arbor, Mich.) 1946.
8. Kuhn, Paul, and Hardrath, Herbert F.: An Engineering Method for Estimating Notch-Size Effect in Fatigue Tests on Steel. NACA TN 2805, 1952.

TABLE 1.- STATIC-STRENGTH PROPERTIES OF SHEET SPECIMENS

[Data for unnotched specimens taken from reference 1]

Specimen type	Average tensile properties			Comparative yield strength, psi
	Elongation in 2 in., percent	Yield strength for 0.2-percent offset, psi	Ultimate strength, psi	
2024-T3 aluminum alloy				
Unnotched	18.2	54,000	73,000	44,500
Notched ( $K_t = 4.0$ )				
$r = 0.004$ in.	----	-----	61,500	-----
$r = 0.070$ in.	----	-----	69,500	-----
7075-T6 aluminum alloy				
Unnotched	11.4	76,000	82,500	74,000
Notched ( $K_t = 4.0$ )				
$r = 0.004$ in.	----	-----	76,200	-----
$r = 0.070$ in.	----	-----	80,000	-----
Normalized SAE 4130 steel				
Unnotched	14.3	98,500	117,000	86,000
Notched ( $K_t = 4.0$ )				
$r = 0.004$ in.	----	-----	117,000	-----
$r = 0.070$ in.	----	-----	126,800	-----

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR EDGE-NOTCHED  
SHEET SPECIMENS WITH  $K_t = 4.0$

(a) 2024-T3 aluminum alloy

Specimen	Nominal maximum stress, psi	Lifetime, cycles	Remarks (a)
$r = 0.004$ in.			
Nominal mean stress, 0 psi			
A24S1-5	32,500	8,000	
A21S1-5	26,000	30,000	
A19S1-6	21,000	104,000	
A19S1-5	16,000	379,000	
A20S1-5	14,000	760,000	
A18S1-6	12,500	1,096,000	
A24S1-4	11,250	1,287,000	
A25S1-5	11,250	11,390,000	
A25S1-4	10,700	11,226,000+	Did not fail
A25S1-3	10,000	11,281,000+	Do.
A20S1-6	10,000	13,420,000+	Do.
Nominal mean stress, 20,000 psi			
A25S1-1	38,000	10,000	
A24S1-6	35,000	18,000	
A21S1-6	32,000	38,500	
A22S1-2	28,500	114,000	
A22S1-1	25,000	523,000	
A23S1-3	22,500	10,475,000+	Did not fail
$r = 0.070$ in.			
Nominal mean stress, 0 psi			
A20S1-3	20,000	5,000	
A19S1-3	17,500	8,000	
A20S1-4	15,000	25,000	
A18S1-2	12,500	56,000	
A23S1-1	10,000	427,000	
A20S1-1	10,000	520,000	
A23S1-2	8,000	2,045,000	
A19S1-1	8,000	3,100,000	
A21S1-2	7,500	3,656,000	
A22S1-6	7,000	2,487,000	
A22S1-5	7,000	5,200,000	
A21S1-3	6,500	17,599,000+	Did not fail
Nominal mean stress, 20,000 psi			
A20S1-2	35,000	8,000	
A21S1-4	31,250	16,800	
A18S1-1	27,500	66,000	
A19S1-2	25,000	201,000	
A22S1-4	24,000	10,183,000+	Did not fail
A22S1-3	23,750	13,223,000+	Do.
A19S1-4	22,500	11,570,000+	Do.

<sup>a</sup>Unless otherwise noted, specimen failed at notch root.

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR EDGE-NOTCHED

SHEET SPECIMENS WITH  $K_t = 4.0$  - Continued

(b) 7075-T6 aluminum alloy

Specimen	Nominal maximum stress, psi	Lifetime, cycles	Remarks (a)	
r = 0.004 in.				
Nominal mean stress, 0 psi				
B28S1-2	35,000	11,000		
B24S1-5	26,000	31,500		
B22S1-6	22,000	93,000		
B23S1-6	17,000	217,000		
B22S1-5	14,000	353,000		
B21S1-6	13,000	482,000		
B23S1-5	10,000	3,375,000		
B24S1-6	9,000	5,045,000		
B28S1-4	8,700	11,299,000+		
B27S1-2	8,000	14,874,000+		
Did not fail Do.				
Nominal mean stress, 20,000 psi				
B28S1-3	40,000	6,300		
B27S1-3	36,000	16,000		
B25S1-5	32,000	26,000		
B26S1-4	28,500	39,500		
B26S1-5	26,750	149,000		
B25S1-6	25,000	2,441,000		
B26S1-6	23,000	14,390,000+		
Did not fail				
r = 0.070 in.				
Nominal mean stress, 0 psi				
B24S1-4	20,000	6,000		
B22S1-4	15,000	26,000		
B23S1-2	12,500	75,500		
B26S1-2	10,000	257,000		
B22S1-2	10,000	307,000		
B23S1-1	8,500	275,000		
B26S1-1	8,000	1,920,000		
B25S1-3	8,000	2,690,000		
B21S1-1	7,000	5,290,000		
B25S1-4	7,000	15,715,000+		
B25S1-1	6,500	11,878,000+		
B21S1-2	6,000	16,758,000+		
Did not fail Do. Do.				
Nominal mean stress, 20,000 psi				
B24S1-1	35,000	5,000		
B24S1-3	31,250	9,700		
B22S1-1	27,500	23,000		
B23S1-4	26,000	35,000		
B21S1-4	23,750	438,000		
B23S1-3	23,125	1,540,000		
B24S1-2	23,000	10,506,000+		
B21S1-3	22,500	16,490,000+		
Did not fail Do.				

<sup>a</sup>Unless otherwise noted, specimen failed at notch root.

TABLE 2.- AXIAL-LOAD FATIGUE TEST RESULTS FOR EDGE-NOTCHED

SHEET SPECIMENS WITH  $K_t = 4.0$  - Concluded

(c) Normalized SAE 4130 steel

Specimen	Nominal maximum stress, psi	Lifetime, cycles	Remarks (a)
$r = 0.004$ in.			
Nominal mean stress, 0 psi			
C22N1-6	43,750	124,000	Did not fail
C21M-6	40,000	215,000	
C23N1-5	35,000	379,000	
C25N1-6	30,000	919,000	
C26N1-4	25,000	1,546,000	
C27N1-1	22,500	3,193,000	
C26N1-5	20,000	14,394,000+	
Nominal mean stress, 20,000 psi			
C24N1-6	65,000	46,000	Did not fail Do.
C21M1-5	60,000	71,000	
C24N1-5	50,000	182,000	
C22N1-5	45,000	347,000	
C23N1-6	42,500	195,000	
C22N1-2	42,500	874,000	
C25N1-5	40,000	389,000	
C27N1-4	40,000	12,508,000+	
C26N1-3	37,500	10,950,000+	
$r = 0.70$ in.			
Nominal mean stress, 0 psi			
C22N1-3	42,500	5,600	Did not fail Do.
C23N1-4	38,750	11,000	
C23N1-1	35,000	37,000	
C24N1-1	32,000	29,000	
C26N1-2	26,000	104,000	
C24N1-3	22,500	150,000	
C22N1-4	20,000	185,000	
C23N1-2	17,500	587,000	
C22N1-2	15,000	800,000	
C23N1-3	15,000	1,989,000	
C26N1-1	14,000	7,240,000	
C23N1-1	13,750	10,910,000+	
C22N1-1	12,500	10,505,000+	
Nominal mean stress, 20,000 psi			
C24N1-4	57,500	9,800	Did not fail
C24N1-2	50,000	24,000	
C21M1-4	39,000	74,000	
C21M1-1	33,000	297,000	
C25N1-4	32,000	1,064,000	
C21N1-3	32,000	11,368,000+	

<sup>a</sup>Unless otherwise noted, specimen failed at notch root.

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TABLE 3.- SUMMARY OF RESULTS OF AXIAL-LOAD FATIGUE TESTS  
ON EDGE-NOTCHED SHEET SPECIMENS WITH  $K_t = 4.0$

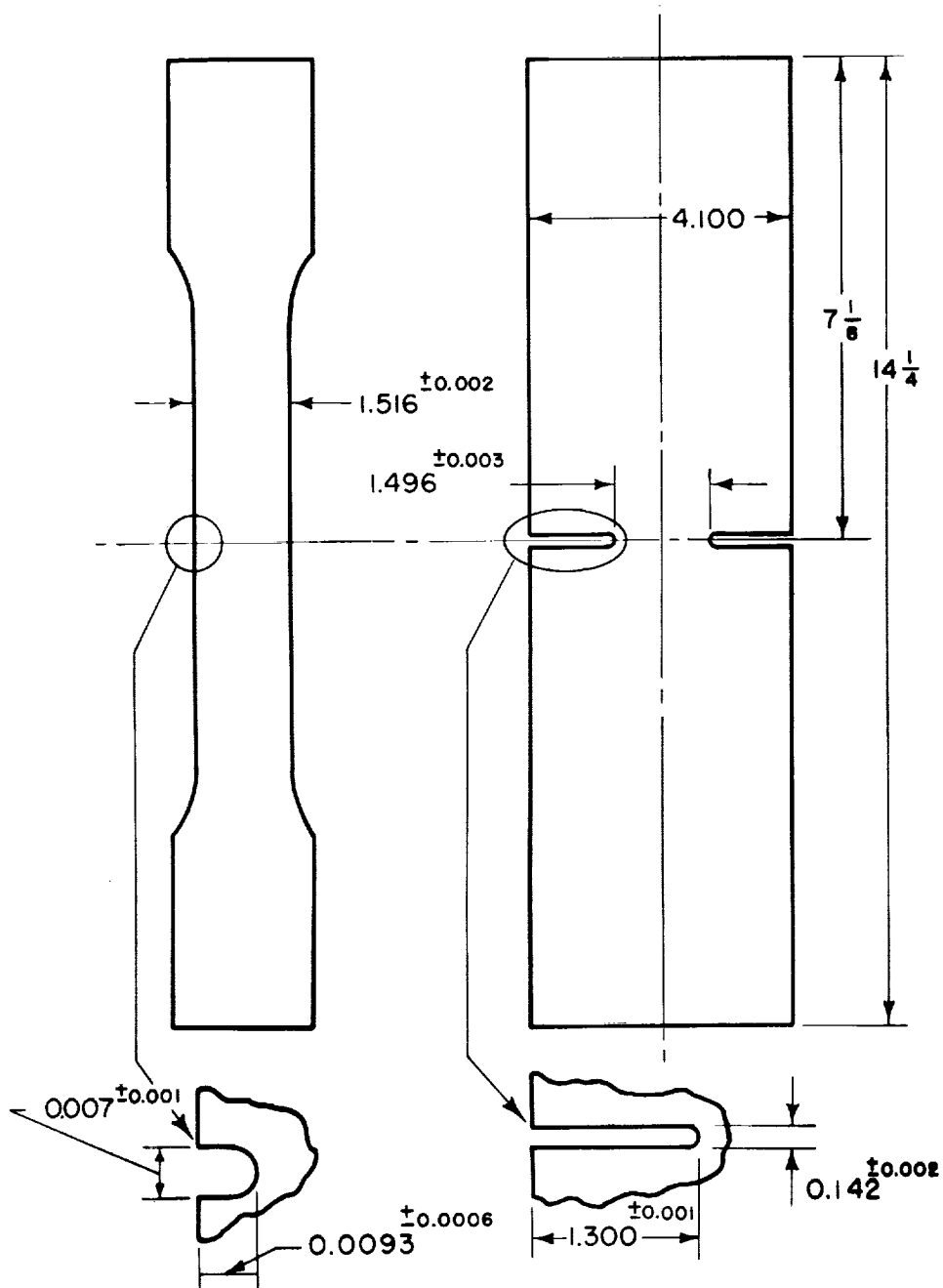
Nominal mean stress, psi	Notch radius, r, in.	Nominal maximum stress, psi, at lifetime, cycles, of -					
		$10^4$	$5 \times 10^4$	$10^5$	$5 \times 10^5$	$10^6$	
2024-T3 aluminum alloy							
0	0.004 a.057 .070	$31.5 \times 10^3$ 18.0 17.5	$24.0 \times 10^3$ 13.0 13.0	$21.0 \times 10^3$ 11.5 11.5	$14.5 \times 10^3$ 8.5 9.0	$12.5 \times 10^3$ 8.0 8.5	$10.5 \times 10^3$ 7.0 7.0
$20 \times 10^3$	.004 a.057 .070	37.5 32.0 33.5	31.0 27.0 27.5	28.5 25.0 26.5	25.5 25.0 24.5	24.5 24.0 23.5	23.0 24.0 23.5
7075-T6 aluminum alloy							
0	0.004 a.057 .070	$34.0 \times 10^3$ 17.0 18.0	$24.5 \times 10^3$ 13.0 14.5	$21.0 \times 10^3$ 11.0 11.5	$15.0 \times 10^3$ 8.5 9.0	$11.0 \times 10^3$ 8.0 8.5	$9.0 \times 10^3$ 7.5 7.0
$20 \times 10^3$	.004 .057 .070	38.5 31.0 31.5	29.0 25.0 26.0	27.5 24.0 25.0	26.0 23.0 24.0	25.5 23.0 23.5	24.0 23.0 23.0
Normalized SAE 4130 steel							
0	0.004 a.057 .070	----- $45.0 \times 10^3$ 39.5	----- $32.0 \times 10^3$ 29.5	----- 27.0 25.0	$33.5 \times 10^3$ 19.0 17.0	$28.5 \times 10^3$ 16.0 15.5	$20.5 \times 10^3$ 14.0 13.5
$20 \times 10^3$	.004 a.057 .070	----- 58.0 57.5	63.5 45.0 42.5	56.5 41.0 37.5	42.0 34.0 32.5	39.5 34.0 32.0	37.5 33.0 31.5

<sup>a</sup>Data for 0.057-in. notch radius taken from ref. 2.

TABLE 4.- FATIGUE-STRENGTH REDUCTION FACTORS FOR  
EDGE-NOTCHED SHEET SPECIMENS WITH  $K_t = 4.0$

Nominal mean stress, psi	Notch radius, r, in.	$K_f$ at lifetime, cycles, of -			
		$10^4$	$10^5$	$10^6$	$10^7$
2024-T3 aluminum alloy					
0	0.004	1.70	1.70	2.00	2.00
	.057	3.00	3.15	3.15	3.00
	.070	3.10	3.15	2.95	3.00
$20 \times 10^3$	.004	----	2.20	2.65	2.90
	.057	----	2.70	2.75	2.75
	.070	----	2.45	2.75	2.80
7075-T6 aluminum alloy					
0	0.004	1.60	1.70	2.25	2.35
	.057	3.15	3.25	3.10	2.80
	.070	3.00	3.15	2.95	3.00
$20 \times 10^3$	.004	----	2.20	2.60	2.75
	.057	----	2.90	3.10	3.05
	.070	----	2.70	2.95	3.05
Normalized SAE 4130 steel					
0	0.004	----	1.45	1.80	2.35
	.057	1.30	2.40	3.25	3.40
	.070	1.30	2.60	3.35	3.55
$20 \times 10^3$	.004	----	1.60	2.20	2.35
	.057	----	2.65	2.85	2.90
	.070	----	2.90	3.10	3.20

W  
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2  
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(a) Edge-notched specimen.

(b) Notched specimen.

Figure 1.- Notch details of notched and edge-notched fatigue test specimens.  $K_t = 4.0$ .



W-125

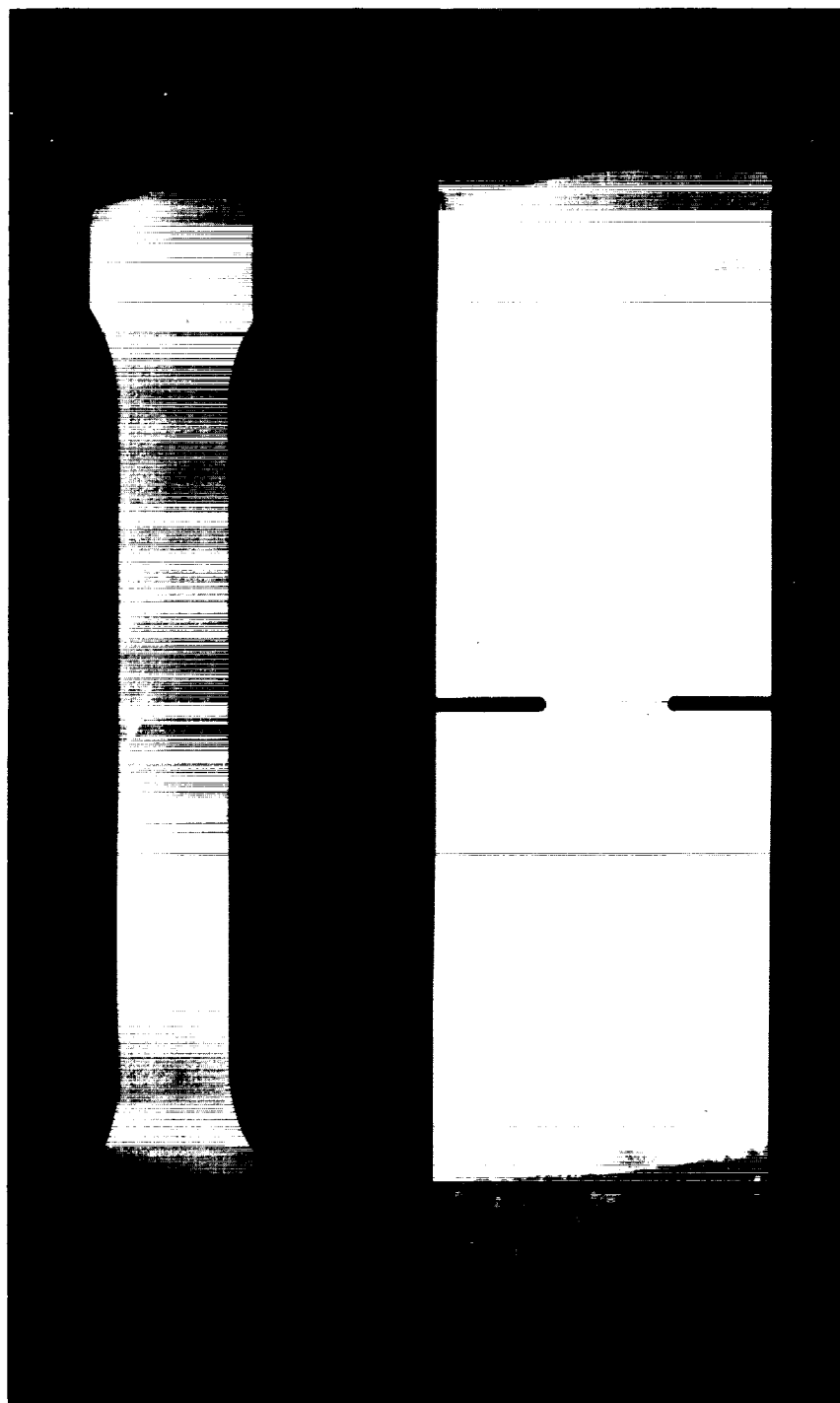


Figure 2.- Typical fatigue failures of notched specimens. L-59-3030

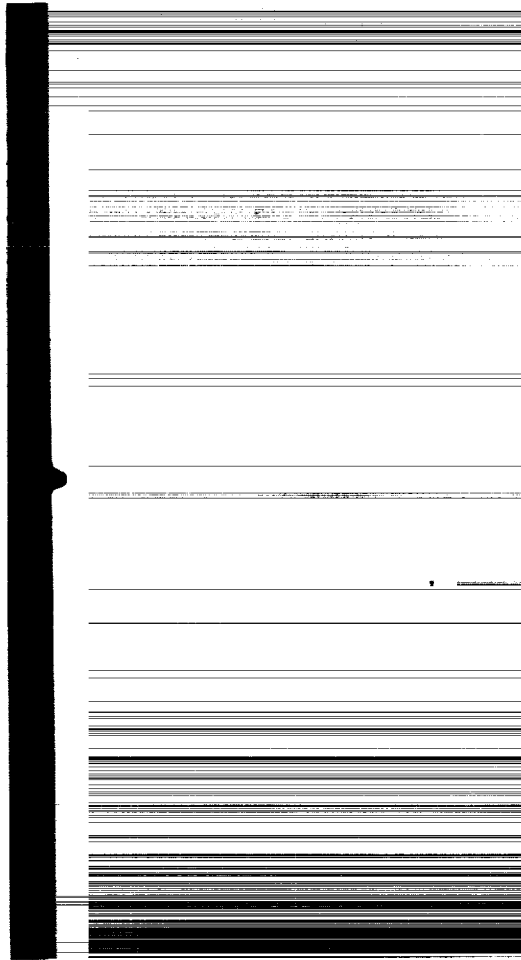
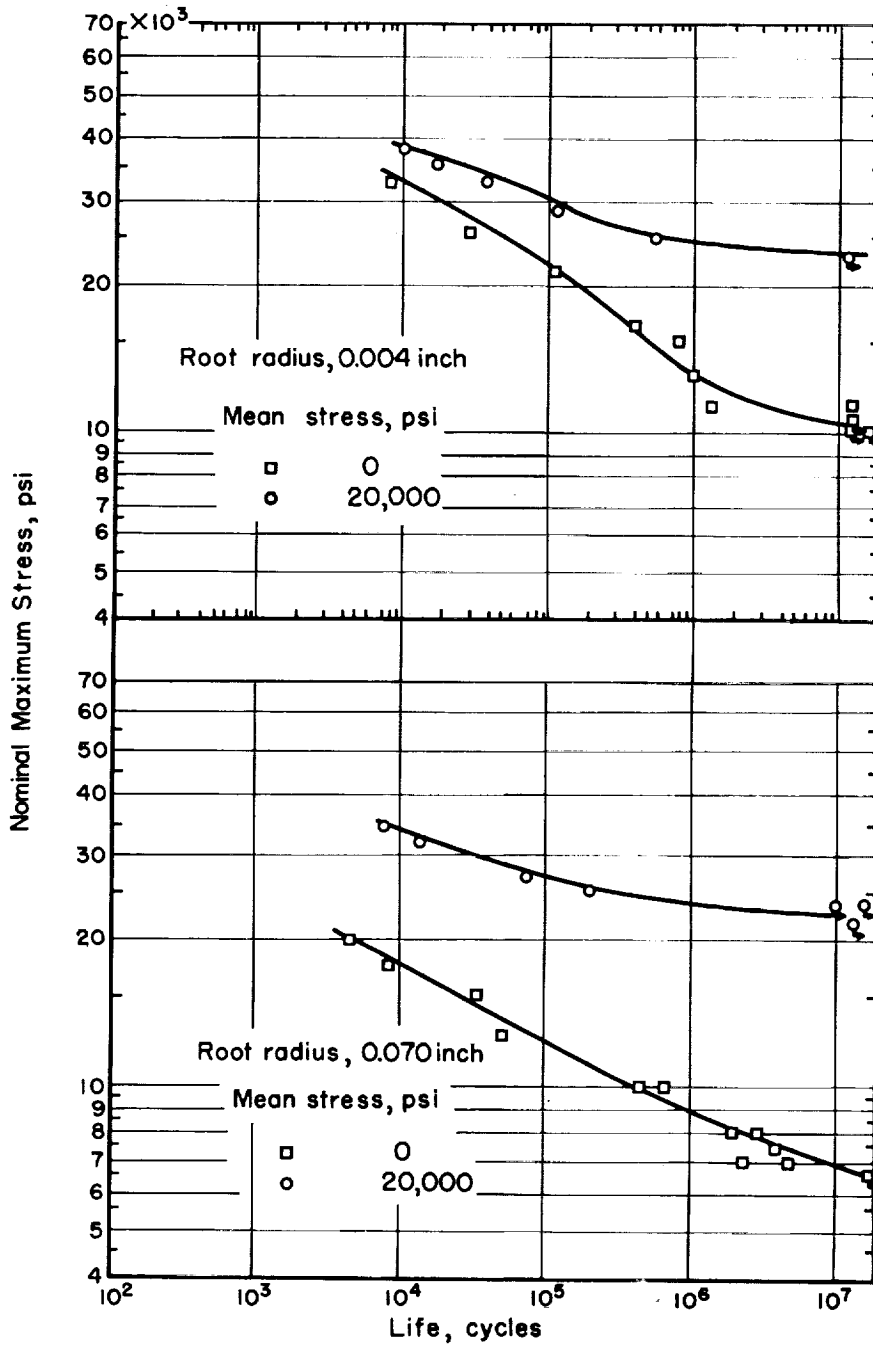


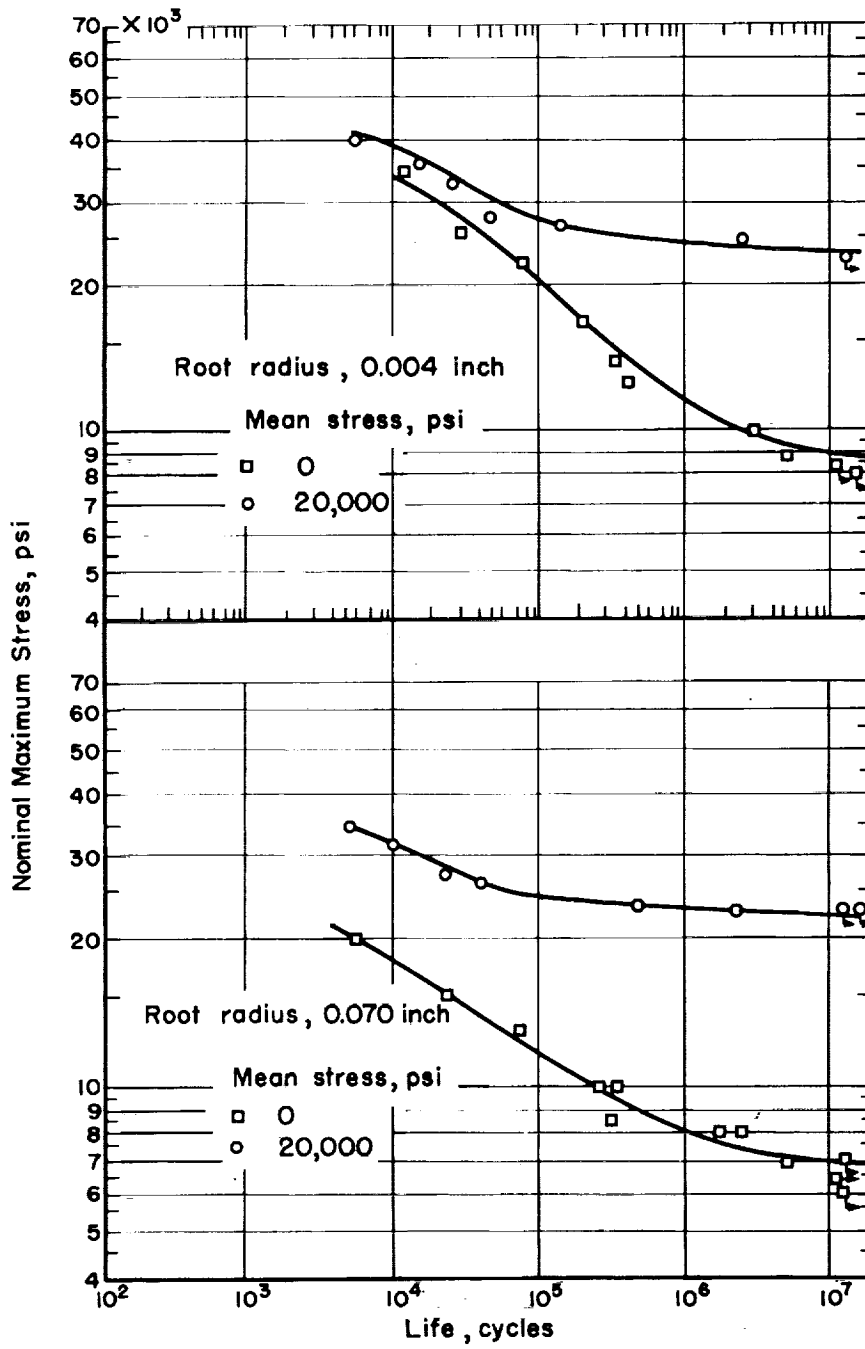
Figure 3.- Fatigue failure of edge-notched specimen.  $K_t = 4.0$ ; root radius, 0.004 inch. L-59-3031

W-125



(a) 2024-T3 aluminum alloy.

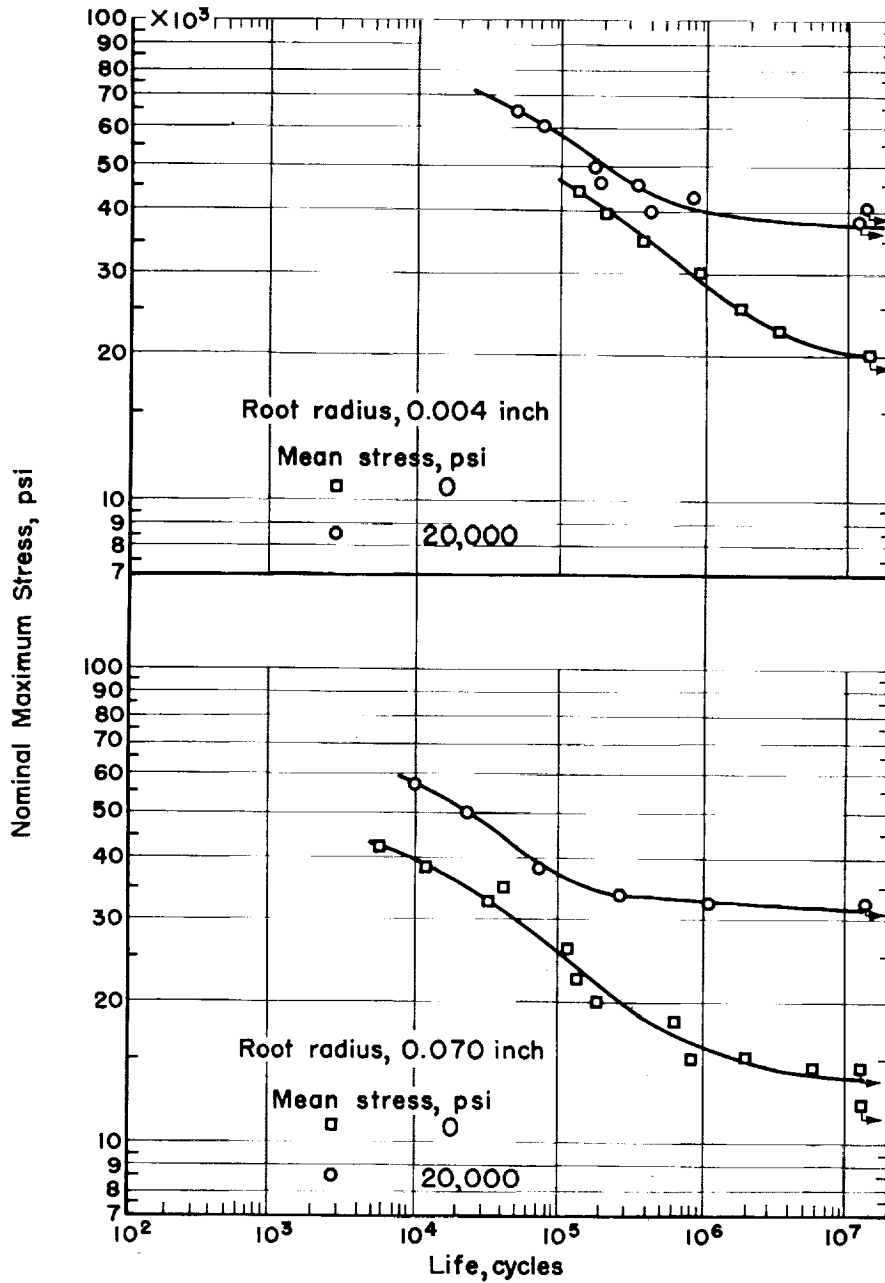
Figure 4.- Results of axial-load fatigue tests on edge-notched sheet specimens.  $K_t = 4.0$ .



(b) 7075-T6 aluminum alloy.

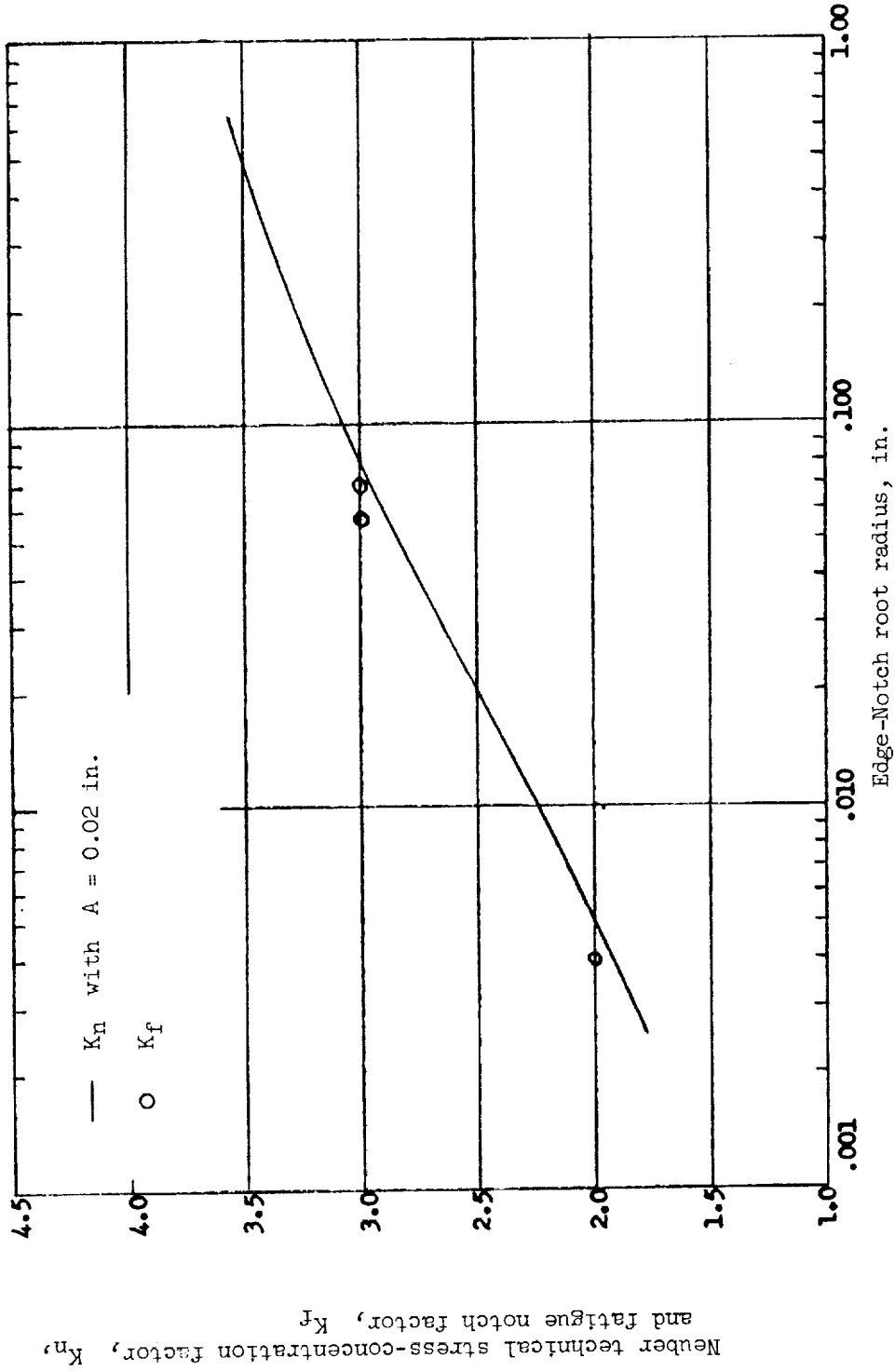
Figure 4.- Continued.

W-125



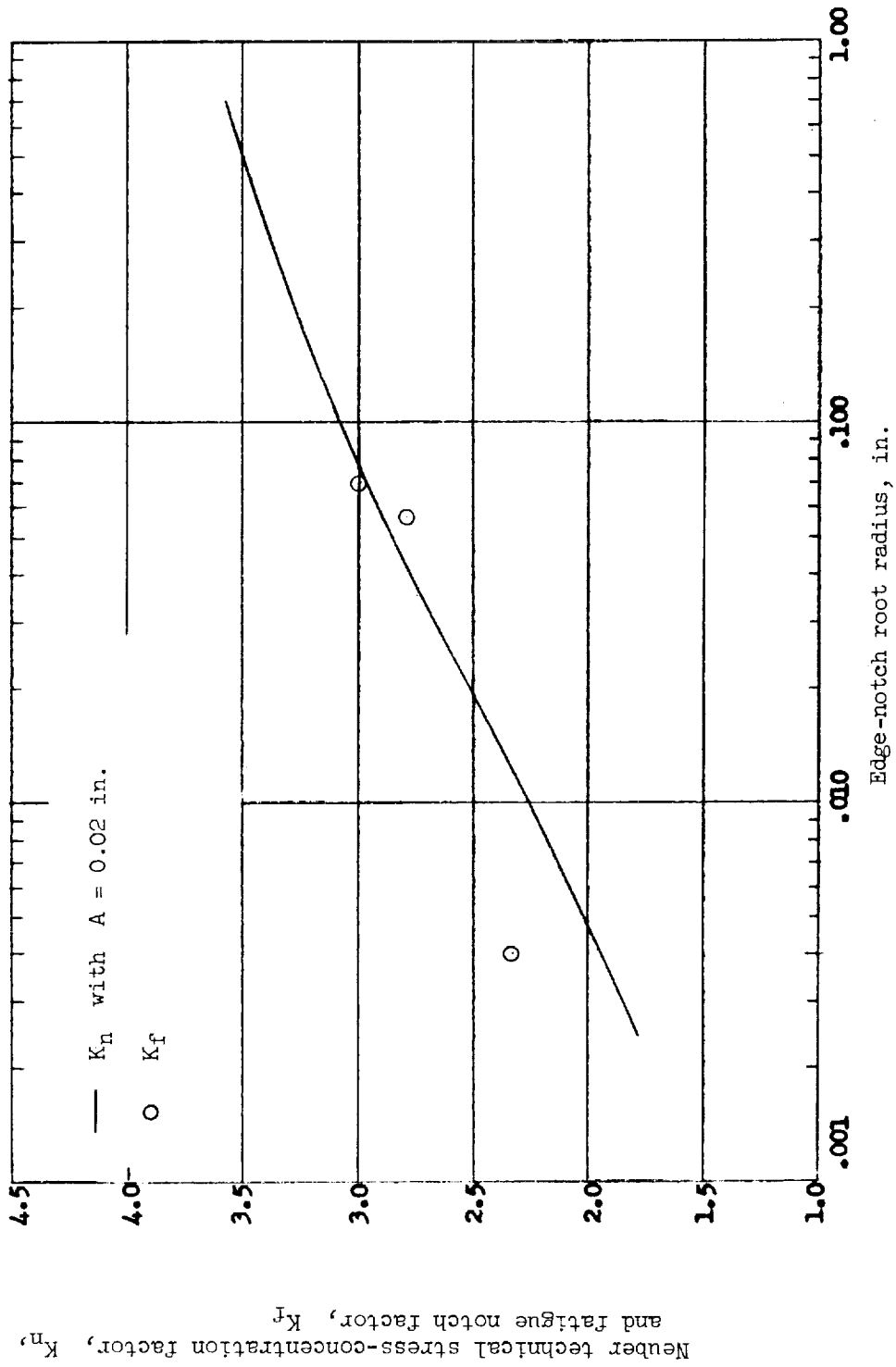
(c) Normalized SAE 4130 steel.

Figure 4.- Concluded.



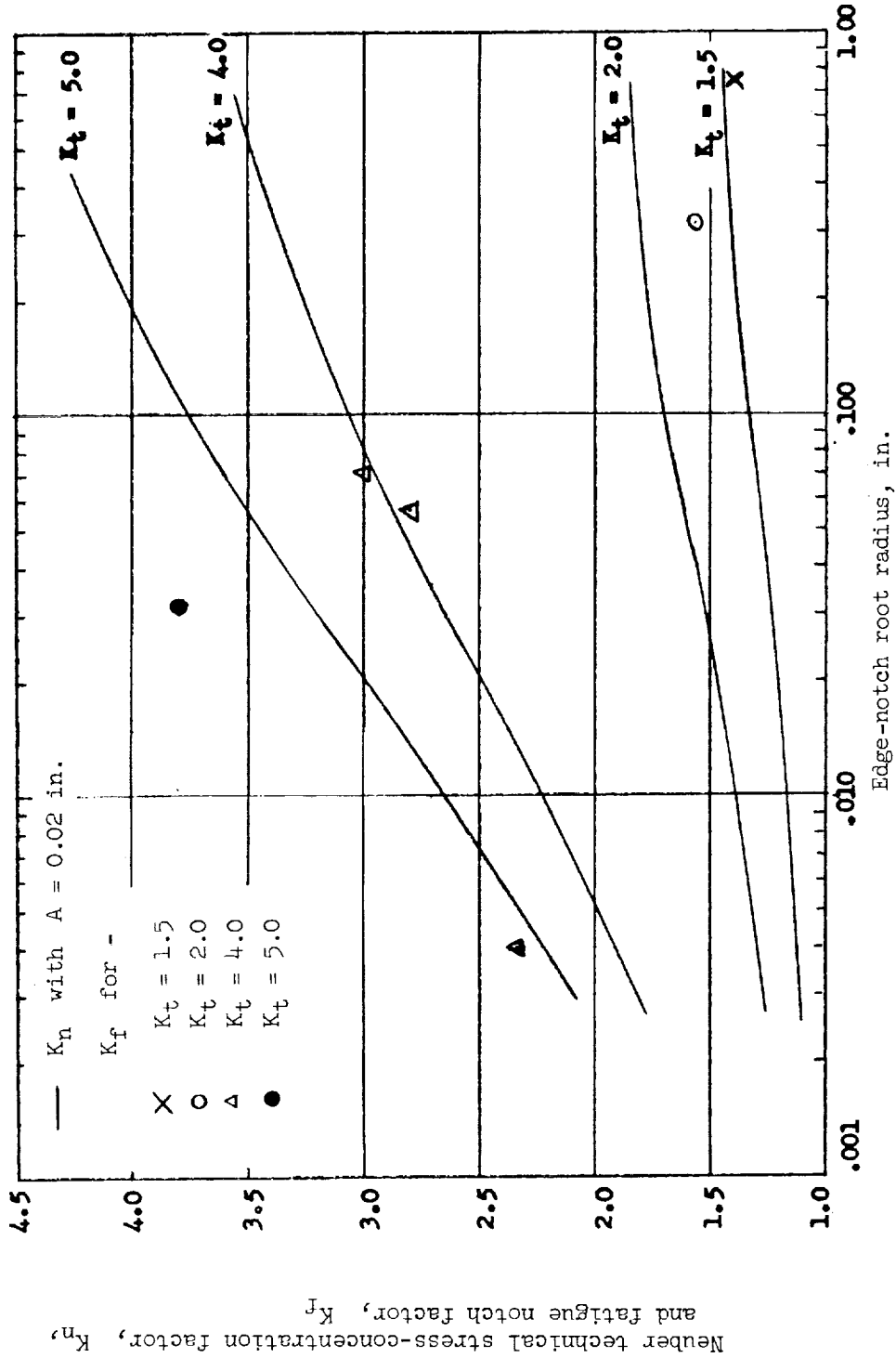
(a) 2024-T3 aluminum alloy.

Figure 5.- Comparison of  $K_f$  with  $K_n$  for specimens having edge notches of various root radii.  
 $K_t = 4.0$ ; stress is reversed.



(b) 7075-T6 aluminum alloy.

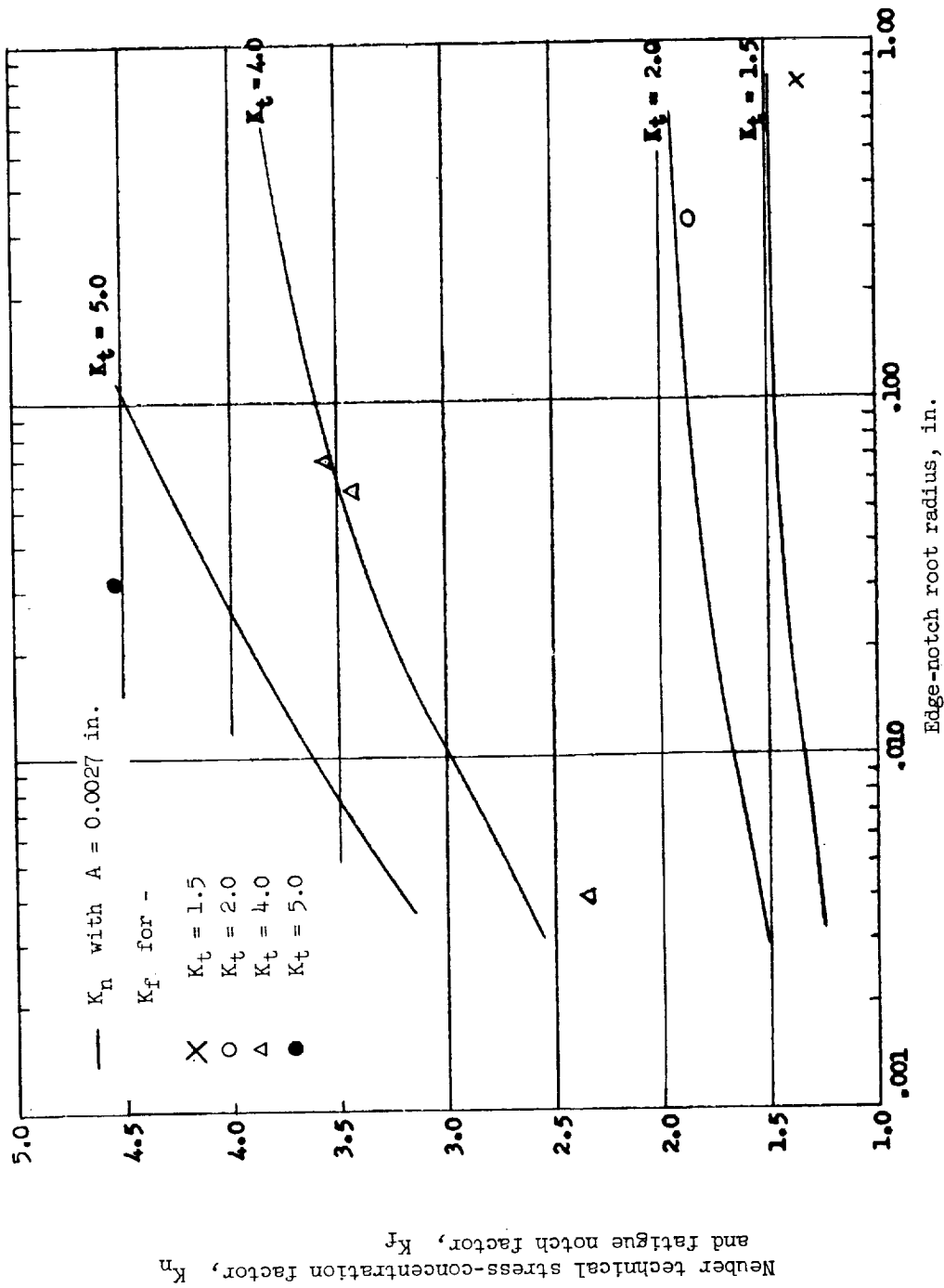
Figure 5.- Continued.



(b) 7075-T6 aluminum alloy.

Figure 6.- Continued.





(c) Normalized SAE 4130 steel.

Figure 6.- Concluded.

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