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# Allison EDR-17923



# Design of a Low Speed Fan Stage for Noise Suppression

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#### SUMMARY

This report describes the design of a low tip speed, moderate pressure rise fan stage for demonstration of noise reduction concepts. The fan rotor is a fixed-pitch configuration delivering a design pressure ratio of 1.378 at a specific flow of 43.1 lbm/sec/ft<sup>2</sup>. Four exit stator configurations were provided to demonstrate the effectiveness of circumferential and axial sweep in reducing rotor-stator interaction tone noise. The fan stage design was combined with an axisymmetric inlet, conical convergent nozzle, and nacelle to form a powered fan-nacelle subscale model. This model has a 22-inch cylindrical flow path and employs a rotor with a 0.30 hub-to-tip radius ratio. The design is fully compatible with an existing NASA force balance and rig drive system.

The stage aerodynamic and structural design is described in detail. Three-dimensional (3-D) computational fluid dynamics (CFD) tools were used to define optimum airfoil sections for both the rotor and stators. A fan tone noise predictive system developed by Pratt & Whitney under contract to NASA was used to determine the acoustic characteristics of the various stator configurations. Parameters varied included rotor-to-stator spacing and vane leading edge sweep. The structural analysis of the rotor and stator are described herein. An integral blade and disk configuration was selected for the rotor. Analysis confirmed adequate low cycle fatigue life, vibratory endurance strength, and aeroelastic suitability. A unique load carrying stator arrangement was selected to minimize generation of tonal noise due to sources other than rotor-stator interaction. Analysis of all static structural components demonstrated adequate strength, fatigue life, and vibratory characteristics.

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#### **1.0 INTRODUCTION**

Since the late 1960s, there has been a continuous effort to lower community noise levels resulting from aircraft terminal operations. Current high bypass ratio engine technology is sufficient to allow certification of aircraft to Stage 3 of Federal Aviation Regulation (FAR) Part 36. As part of the natural evolutionary process, consideration of a reduced certification level is underway. In order to accommodate a growth plan, major reductions in propulsion system generated noise will be required for new aircraft/ engine combinations to be certified to this more stringent noise standard. Under Task 5 of contract NAS3-25950, Allison Engine Company studied the engine component noise reductions required to produce a propulsion system for a twin engine aircraft producing certification levels 10 decibels (dB). below the current FAR 36 Stage 3 requirement. Early results of this study indicated a strong acoustic advantage in moving from a conventional six bypass ratio turbofan cycle to an ultrahigh bypass ratio cycle employing a low pressure ratio, low tip speed fan. However, cycle changes alone were not sufficient to produce flyover levels 10 dB below stage 3. Additional reductions required identification of innovative strategies for lowering the strength of dominant noise sources. Flyover time histories of perceived noise level produced under this contract indicated the predominant noise source was the fan during both the takeoff and approach segments of flight. Noise reduction studies based on this result identified bypass vane sweep as a potentially effective approach for reducing the pure tone portion of the fan noise field. Based on the results of these studies, a fan rig test program was proposed to the National Aeronautics and Space Administration (NASA) to demonstrate this concept. As a result of this proposal, a 22-inch diameter singlestage fan demonstrator has been designed. This report documents the aerodynamic and structural design of this stage.

#### 2.0 RIG DESIGN FEATURES

The rig mechanical arrangement evolved from a set of requirements developed to meet the program technical objectives and to satisfy facility and operational needs. Specifically these requirements were:

- the rig must be compatible with existing NASA drive system
- no flow-path obstructions except rotor and stator allowable
- provisions must be made for multiple vane configurations
- provisions must be must be accomplished in the wind tunnel and not require removal of fan rotor

The final configuration, shown in Figure 1, meets all design objectives and is fully compatible with the NASA drive rig. Based on acoustic analysis, four vane configurations will be tested. Removal of additional flow-path obstructions was required to isolate, as fully as possible, the acoustic impact of the vane geometry changes. As a result of this requirement, the stator must carry not only its normal aerodynamic loads, but also any nacelle generated loads. To accomplish this and allow vane changes without fan rotor removal, the vanes have been designed as a segmented ring with the airfoils providing a load path between flange rings on the inner and outer diameters. Loads are passed from the vane ring to a backbone support though a single shear pin and three radial fasteners in each segment. All outer flow-path pieces aft of the rotor are split axially to allow quick access to the vane fasteners for removal. Multiple attachment planes are provided to accommodate the four vane configurations to be tested. No provisions have been included for either a core rotor or a separate core flow stream.





#### 3.0 AERODYNAMIC DESIGN

## 3.1 FAN STAGE AERODYNAMIC DESIGN

# 3.1.1 Baseline Stage Configuration And Vector Diagrams

The aerodynamic design point for the fan stage, as established during cycle optimization studies conducted during Task 5, is:

Tip Speed = 1000 ft/sec Stage Pressure Ratio = 1.362  $W \sqrt{\theta}/\delta A = 43.1 \text{ lbm/sec/ft}^2$ 

As shown in Figure 2, the final engine configuration of Task 5 employed a booster stage on the fan shaft to provide the required supercharging for the core compressor. Early in the rig design, it was decided both the booster stage and the core flow bifurcation would be eliminated. This produced two benefits. The first was a significant reduction in mechanical complexity, resulting in reduced fabrication costs. The second was the removal of additional noise sources, allowing a clear identification of the acoustic benefit of vane geometry variations. As a result of the very high bypass ratio cycle selected in the Task 5 engine study, a strong radial rotor exit total pressure gradient exists (Figure 3). This profile is also present in the rig design. The rig stage design pressure ratio was selected as the mass average of the 1.38 bypass and 1.21 core pressure of the original engine design, allowing for some loss through the rig stator. A schematic cross section of the baseline rig configuration is shown in Figure 4. As can be seen, a cylindrical outer flow-path contour was maintained through the stator exit. The requisite area ruling through the stage is introduced through the hub flow path as an integral part of the blading design. Curvature was used into and through the stator to keep the relatively low momentum fluid coming from the rotor hub energized. The rotor-to-stator axial gap is consistent with current Allison fans. Coordinates for the flow path of the baseline configuration are presented in Table 1.

The velocity vector diagrams were generated using the Allison axisymmetric streamline curvature design system. A listing for the aerodynamic design point is included in Appendix A. Some of the blade and vane inlet and exit profiles tabulated in Appendix A are plotted in Figures 5 and 6 Also shown are corresponding profiles from the NASA Stage 53 fan. The comparison is useful since the general character of the flow field through the two fans is similar. The NASA Stage 53 fan was designed for the same rotor pressure ratio and tip speed; it did not quite pump to design intent, hence, the profiles measured at design flow are shown in addition to those that represent design intent. The low noise fan (LNF) rotor is designed for a pressure profile of even greater skew and for higher throughflow velocities than found in the NASA Stage 53 fan. The rotor inlet is also set for a higher specific flow and lower inlet radius ratio (0.30). As a result, the inlet relative Mach number at the tip for the LNF is higher, 1.143, even though tip speeds are the same. Greater turning is required across the LNF blade tip but the blade is overall more lightly loaded.

Velocities at stator inlet, although subsonic, are relatively high toward the outer diameter due to the pressure profile from the rotor and the absence of a splitter. This, together with the thicknesses and camber required of these vane sections, made it impossible to design an entirely shock-free stator. Stator loading was reduced and performance enhanced by allowing closure of the discharge annulus to a Mach number of 0.59 (including blockage). The turning required through the baseline vane row is thus considerably less than was required through the NASA 53 fan stator.

#### 3.1.2 Blade Design

The fan blade was designed as if destined for a commercial fan application to ensure as much realism was incorporated in its geometry as possible. The ultrahigh bypass ratio engine preliminary design cycle was



Figure 2. Final engine configuration - Task 5.



Figure 3. Rotor exit total pressure profile.



Figure 4. Baseline low noise fan schematic (meridional view).

#### Table I. <u>Flow-path coordinates for LNFB.</u>

Tip contour is a straigh line of radius 11.00000 in.

	Z	R	
<b>C</b> ·			-
Spinner nose	-5.7500	0.0000	
	-5.5000	0.211325	
	-5.0000	0.5000	— straight line
	-1.5000	2.520756	- segment*
<b>-</b>	-1.0000	2.8000	orginein
Rotor LE	0.0000	3.3000	
	0.4000	3.4770	
	1.0000	3.6700	
	1.4000	3.7700	
	1.8000	3.8900	stack axis = 1 5584
<b>—</b> —	2.2000	4.0400	
Rotor TE	3.2940	4.4330	
	4.0000	4.6350	
	5.0000	4.8450	
	5.6000	4.9420	
	5.8000	4.9750	
	6.0000	4.9850	
Chaine T.D.	7.0000	5.1200	
Stator LE	7.4120	5.1930	
	7.8000	5.3000	
	8.2000	5.4100	stack axis $\equiv 8.2518$
	8.6000	5.5500	
Stator IE	9.0930	5.6000	
	10.0000	5.6000	— straight line
	11.0000	5.6000	- segment
* $R = mZ + b$			
where: m = tan 30 deg			
B = 3.3867513			

assumed, including an 85% speed takeoff condition, so part-speed performance could be considered. Analytically, the blade demonstrates over 16% surge margin at design speed. Leading edge thickness (Figure 7) was selected consistent with current bird strike criteria. Trailing edge thickness was set equal to leading edge thickness everywhere except near the hub, where a blunter leading edge was employed to improve the hub inlet flow field. Blade chord varies linearly such that the tip is 45% longer than the hub. The spanwise distribution of maximum thickness-to-chord is also shov/n and ranges from 2.75% at the tip to 9.42% at the hub. The locations of maximum thickness for each section (not shown) were shifted from a uniform 50% chord to improve passage area qualities. Geometric properties are tabulated in Appendix A.



Figure 5. Low noise fan rotor design point profiles (1 of 2).



Figure 5. Low noise fan rotor design point profiles (2 of 2).



Figure 6. Low noise fan baseline stator design point profiles (1 of 2).



Figure 6. Low noise fan baseline stator design poir t profiles (2 of 2).

Preliminary design of the blading was carried out assuming multiple-circular-arc (MCA) airfoil sections. Given the low tip speed of this fan, an MCA blade was acceptable for studying the effects of changes in aspect ratio, maximum thickness, and spanwise chord distributions on surge margin and mechanical integrity. The final blade is made up of sections of aerodynamically-optimized meanlines with nearsinusoidal thickness distributions. Viscous computational analysis was used extensively to obtain the desired match of the blade passages with the design intent flow field. The transonic sections were tailored for the design speed shock structure permitting the largest excursion in flow range to stall with acceptable performance.

The spanwise distribution of incidence angles to which the blade sections were set, shown as the solid line in Figure 8 evolved from several considerations. One was the decision to design to relatively tight throats (3.5% throat margin) to favor operating line performance. In the portion of the blade with supersonic inlet relative flow, another consideration was to observe the first captured Mach wave rule. This is a ruleof-thumb setting a critical incidence off the suction surface at a point halfway between the leading edge and the point of emanation of the first captured Mach wave to a minimum of 1.5 degrees, to ensure flowhandling capability. A third consideration involved the meanlines of all sections which were carefully shaped to produce acceptable surface Mach number distributions devoid of local peaks or spikes. This could be done over the outer half of the blade only by straightening the meanlines forward of the throat locations and forcing the bulk of the turning aft (Figure 9). Where possible, the subsonic sections were tailored for shock-free (design point) operation. Optimum chordwise loading distributions were achieved by keeping meanline curvature well forward and closing the leading edge. All this led to incidences considerably smaller than employed in the design of the NASA Stage 53 rotor.

The predicted surface distributions of isentropic Mach number and associated passage Mach number contours for the near-tip, pitch, and near-hub sections of the blade are shown in Figures 10, 11, and 12. The near-tip section was fashioned to produce a single, oblique shock pulled well back into the passage and impinging on the suction surface just ahead of the region of greatest curvature. The suction surface Mach number rises smoothly to a peak of about 1.35. The pitch section, shown in Figure 11, was shaped to operate shock free. Maximum thickness was brought forward to the mouth and curvature was distributed over a larger portion of the section to flatten the forward portion of the suction surface velocity distribution.

Area-ruling of the hub flow path was an integral part of the design of the near-hub sections. Due to thickness, the hub was found to be quite insensitive to incidence and local meanline changes. Modification of the hub flow path improved the loading distributions. The intent was to force the section loading forward without allowing the hub to overpump (due to greatly increased camber). Several iterations were required, with the final outcome shown in Figure 12.

The rotor deviation angles, shown in Figure 8 were set by augmenting calculated NASA 2-D rule deviations with the empirically-estimated corrections plotted in Figure 13. These corrections have been established through comparisons of computational and measured results from other Allison compressor stages, as well as published reports. The computational results suggest, for the deviation distribution chosen, there is sufficient camber in the blading to produce the desired pressure profile. The velocity vectors for the near-tip, pitch, and near-hub sections reveal a healthy flow field with no trace of incipient separation (Figure 14).

The static or manufactured blade geometry producing the desired blade shape at design speed was determined by subtracting the predicted deflection of the blade due to centrifugal and aerodynamic loading



Figure 7. Rotor blade geometric parameters.



Figure 8. Incidence and deviation angles (degrees).

applied at the design point. These deflections were determined using an Allison proprietary finite element structural analysis procedure. Airfoil sections are defined on planes normal to the stack axis. The stack axis is a radial line passing through the center of gravity of each conical section. The leading edge shapes are elliptical. The blading opens with speed by as much as 2 deg in stagger at the tip, due mostly to flexibility of the leading edge. Associated with this movement in the blade-to-blade view, which clearly affects flow handling and pumping capacity, is the radial growth of the tip, with its consequences on clearance effects.

#### 3.1.3 Baseline Fan Vane Design

A view of the baseline stator design, fan configuration No. 1 (FC1), is shown in Figure 14. Unlike the rotor, the stators are unique to the 22-in. NASA rig vehicle because none could be directly scaled-up for use in a high bypass turbofan. In an engine, separate stator assemblies would be required for the bypass and core flow streams. Neither of these assemblies would necessarily reproduce a section of the rig stators, due to the presence of the flow splitter. Nevertheless, the stators are crucial components of the rig tests. The baseline stator must deliver the same performance and allow no more noise in the acoustic test vehicle than would the bypass stator in a representative commercial turbofan.

The dominant feature of the stator flow field is its nonuniform, high-velocity inlet (Figure 6). The baseline stator is relatively lightly loaded and does not have to affect a large amount of turning, so the emphasis during its design was on minimizing total pressure loss. As a result, the vane design process primarily involved the selection of an incidence distribution. For any given incidence, neither meanline shape, maximum thickness, nor section thickness distribution had any appreciable effect on performance. Therefore, simple double circular arc sections with maximum thickness located just forward of mid-chord were employed.



Figure 9. Blade near-tip Mach number distribution.



Figure 10. Blade pitch section Mach number distribution.



Figure 11. Blade near-hub Mach number distribution.



Figure 12. Empirical modifications to the rotor deviation profile.



Figure 13. Rotor passage velocity vectors.



Figure 14. Baseline vane arrangement.

A wide range of incidence levels were examined in an effort to optimize velocity distributions about the vane sections and to minimize suction surface velocity peaks, but it became apparent there were basically only two solutions. These solutions are represented in Figure 15 by the "A" and "B" incidence distributions. The leading edge (at design flow) could either be A) optimally aligned with or set closed relative to the incoming flow, which invariably produced supersonic velocities over the forward third of the outer sections or B) set open relative to the incoming flow to produce velocity distributions with reduced trailing edge loading. All attempts to combine the two types radially forced the outer sections toward "A"-type distributions.

A "B"-type design was finally chosen for FC1. The design offered reduced suction surface Mach number peaks in exchange for increased leading edge loading. It was felt the more open leading edges would not be a liability in this stage, given the large axial gap between the rotor and stator. Deviation was reduced for the "B"-type vane, as shown in Figure 15, while throats were not excessive. The deviation angle profile was adjusted to remove all swirl as would be required of a bypass stator.

The surface isentropic Mach number distributions and associated passage Mach number contours are shown in Figures 16, 17, and 18 for the near outer diameter, pitch, and near inner diameter "B" vane sections. The inner diameter flow path was contoured through the vane, as it was through the blade, to help balance the loading distributions of the near-hub sections.

The mechanical properties of the baseline vane are tabulated in Appendix A. These properties were retained in designing the alternative vanes. Most have constant spanwise distributions; e.g. maximum thickness-to-chord is 5% and chord is 1.81 in.



Figure 15. Baseline stator incidence, deviation, and throat margin.



Figure 16. Baseline stator near-tip Mach number distribution.



Figure 17. Baseline stator midspan Mach number distribution.



Figure 18. Baseline stator near-hub Mach number distribution.

#### 3.1.4 Fan Stage Analysis

#### 3.1.4.1 Predicted Map

The predicted 100% and 85% speedline characteristics for the low noise fan (LNF) rotor are a composite of analytical and empirical considerations (Figure 19). The shaded circles in the figure represent analysis results at various backpressures for 100% and 85% corrected speed. No attempt was made to include the untwist characteristics of the blade with speed or throttling. To model the indicated aerodynamic design point, the code was run to an "equivalent" design point just over 1% higher in flow and pressure ratio. This was done in light of prior experience with the code to be explained in section 3.1.4.2 below. The background speedlines result from scaling the experimentally-derived map of the NASA Stage 53 rotor to the fan rotor design point and are included for reference to trends only. The speedline scaled from the NASA 53 data roughly corresponds to the computationally predicted behavior of the current design. The design intent surge margin of 15% was obtained. The associated contours of predicted efficiency are also shown with the NASA Stage 53 rotor data, scaled for flow, in the background. These data were more difficult to assess. The computational procedure, at least for high speed machines, typically predicts efficiencies 2 to 3 points higher than are actually attained; this has been assumed a function of computational limits preventing running the code with sufficiently dense grids to accurately reproduce profile drag due to skin friction. Therefore, the predicted efficiency has been modified to better fit the available data. In general, the modified efficiency follows the trends predicted by the code, but reduced at the design condition to correspond to the value obtained from the axisymmetric streamline curvature procedure. Additionally, the rate of efficiency loss beyond the peak has been increased from the computational predictions to mirror the NASA Stage 53 data.

#### 3.1.4.2 Off-Design Performance

The LNF rotor, though part of a research vehicle to be built for acoustics testing, was designed to standards allowing it to be scaled-up directly for use in a large turbofan engine. For that reason, an effort was made to ensure the blade would also demonstrate good off-design performance. It was analyzed along the operating line, near stall, and at an unthrottled condition at both 100% and 85% speed. Figures 20 through 27 show how the LNF rotor is expected to throttle at design speed.

The changes occuring in the total pressure and loss profiles of the rotor with throttling are shown in Figure 20. The long dashed line labeled "ADP-BD76" is the design inten profile from the axisymmetric streamline curvature design code. The three other lines are the profiles predicted by the numerical solution at the three points along the design speedline highlighted in the map of Figure 19. The CFD solution characteristically indicates a stronger hub and a weaker tip than seems to develop in reality, so the profile labeled "ADP-Dawes" was selected as the one to use for the detailed design of the blade. Here again, the analysis of the NASA Stage 53 rotor flow field proved useful. The differences between the BD76 and computational profiles for that machine were considered in establishing the LNF design profile. The unthrottled and near-stall pressure profiles indicate pumping at the hub (which would deliver the core flow in the turbofan) remains unchanged while the bypass portion of the blade, from 20% span to the tip, throttles proportionately with radius. Losses increase with throttling in a consistent manner except, curiously, at the near-tip near stall where they apparently decrease. The changes in throughflow velocities are reflected in the profiles of inlet relative Mach number and air angles (Figure 21). As the blade tip throttles, it maintains flow, while the fraction of flow through the hub decreases. The hub incidence increases 5-6 degrees while the tip increases only 2-3 degrees. The discharge air angles remain little changed over virtually the entire blade span, another indication there is sufficient camber in the blade and the turning can be sustained without a breakdown in the flow field right up to stall.

The predicted changes in surface Mach number distributions and passage Mach number contours for the near-tip, pitch, and near-hub sections with throttling are shown in Figures 22 through 27. Most noticeable



Figure 19. Predicted rotor-only performance map.






Figure 21. Effect of throttling on rotor Mach number and air angles at design speed.



Figure 22. Effect of throttling on blade surface Mach numl er — near tip section.



Figure 23. Effect of throttling on blade passage Mach number — near-tip section.



Figure 24. Effect of throttling on blade surface Mach number -- midspan section.



Figure 25. Effect of throttling on blade passage Mach number — midspan section.



Figure 26. Effect of throttling on blade surface Mach number - near-hub section.



Relative Mach Number, 5.9% Span



Figure 27. Effect of throttling on blade passage Mach number — near-hub section.

is the movement of the shock near the tip. It migrates from within the passage, as an over-expansion normal shock (probably a reflection from the suction surface of a very weak leading edge oblique shock) to a strong, started oblique leading edge shock to an unstarted, though still stable, normal position to a final, (not shown) unstable interaction with the pressure side bow waves from the neighboring blade. It is the ability of 3-D codes to reproduce shock system geometries and reveal the effects on performance of shock structures that make them such powerful design tools. Note the problem of suction side peakiness, discussed earlier, cannot be avoided in the unthrottled condition. The shock is far enough aft that it impinges on the blade in the region of greatest curvature. A vestige of the near-tip flow field can still be seen in the pitch passages, though it was possible to design the pitch section for shock-free operation at the design point. The hub experiences the largest change in flow level and not surprisingly, becomes the pinch point with decreasing backpressure. The surface Mach number distributions illustrate the rationale for the selection of incidences discussed earlier. The progression from negative to design to larger incidences with throttling is apparent.

Once a blade shape acceptable at design speed was defined, it was analyzed at 85% speed, which was defined as takeoff speed in the ultrahigh bypass engine cycle. At this speed, the blade tip inlet runs to just under Mach 1.0. Obviously, the nominal operating line condition at this speed is particularly important from a noise production standpoint. The surface Mach number distributions and passage Mach number contours predicted for the near-tip, pitch, and near-hub sections at the takeoff point are shown in Figures 28, 29, and 30. Notably, even the outermost section operates shock free.

Incidence levels are uniformly higher than at design speed. The near-tip and pitch sections also exhibit a pronounced reacceleration bump in their suction surface Mach number distributions. This is produced by the large local curvature in each section, discussed earlier, that is in turn one consequence of designing to relatively tight throat margins.

### 3.1.5 Additional Vane Designs

The NASA test plan calls for the acoustic evaluation of four distinct configurations. Each is characterized by a different stator; the rotor design described earlier is common to all. The baseline fan includes the radial vane already described. The second configuration of the fan, designated FC2 and shown in Figure 1b, results from repositioning the baseline stator further downstream and increases the rotor-to-stator axial gap. Although the vanes are placed in a slightly different flow field, as modeled in Appendix B, the stator assembly itself remains unchanged. The third and fourth fan configurations, however, necessitated the design of two new stator vanes and associated flow-path modifications. For fan configuration No. 3 (FC3) the stator of the baseline fan is replaced with a vane whose leading edge lies at a 30 degree angle from vertical. The fourth fan, FC4, replaces this stator with another made up of vanes that are both swept and leaned. These latter two stator designs are described below.

### 3.1.5.1 Axially Swept Vane Design

From an acoustic study conducted by NASA, it was determined that among a candidate set of purely swept shapes, a vane swept 30 degrees aft offered the best potential for noise reduction. A stator with this amount of sweep was designed so the radial vane stator of FC1 could be replaced, requiring only one additional spoolpiece to recomplete the outer casing. That the extra spoolpiece was required in any case proved fortuitous since it was found during design of the swept vane that the outer flow path could not be kept of constant radius. A meridional view of the swept vane fan, FC3, is shown in Figure 1c. Not only is the vane highly swept but the casing forward of and through the vane includes a substantial bulge. The incorporations of sweep so increased throughflow velocities that changes in airfoil sections alone were not enough to produce satisfactory outboard vane passage designs; careful area-ruling of the flow-path annulus had to be considered at the same time. Several casing and hub wall contours were analyzed to optimize the final flow-path geometry.







Figure 29. Rotor blade Mach number distribution at simulated takeoff speed — midspan section.



Figure 30. Rotor blade Mach number distribution at simulated takeoff speed — near-hub section.

The design objective was a swept stator with the same kind of velocity distributions over the vane surfaces as were obtained with the baseline vane. All of the physical properties of the baseline vane, i.e. maximum and edge thicknesses, chord, location of maximum thickness, etc, were preserved in both this and the following swept vane designs. Double circular arc sections were employed as before although, due to the sweep, deviation angles increased. Since the outer diameter bulge not only reduced the level but also flattened the shape of the throughflow velocity profile, incidences were adjusted accordingly. Profiles of these parameters are shown in Figure 31 compared with those for the radial vane, *at the respective vane edges*. The surface isentropic Mach number distributions and associated passage Mach number contours are shown in Figures 32, 33, and 34 for the near-tip, midspan, and near-hub sections shown for the baseline vane. A listing detailing FC3 conditions at the aerodynamic design point is included in Appendix C.

# 3.1.5.2 Swept and tangentially Tilted Vane Design

The NASA acoustic study referred to previously indicates a potential for further noise reduction by adding lean (tangential tilt) to a swept vane. The study suggests a vane leaned 30 degrees suction-side down (toward the I.D.) with the lean, like the sweep, incorporated so the vane edges remain straight (viewed along engine centerline) offers the largest benefit. The FC4 vane was designed for this degree of stack axis lean. The final geometry is shown in Figure 1d. Noticeably absent is the large bulge in the O.D. flow path required in FC3. Referring to Figure 35, it can be observed that vane lean increases the flow blockage, producing a proportional increase in throughflow velocity, but tends to reduce the migration of flow toward the outer flow path compared to the simple swept design. As a result, flow-path contouring upstream of the leading edge is not required. Deviation shows a strong sensitivity to loading. In the outboard sections, increased loading produces an increase in inlet-to-discharge velocity ratio; as a result, deviation angle increases. For the inboard sections, increased loading results in a decrease in section velocity ratio; as a result, the deviation angle decreases.

As for FC3, the design objective for the swept and leaned vane was to reproduce velocity distributions over its surfaces as much like those obtained for the radial vane as possible. Section incidences were adapted to help achieve this. The resultant distributions for the usual three sample sections are shown in Figures 36, 37, and 38. A listing detailing FC4 conditions at the aerodynamic design point is included in Appendix D.

### 3.2 NACELLE AERODYNAMIC DESIGN

This nacelle design was developed to meet the basic operational requirements of an isolated nacelle configuration for subsonic/transonic application having an advanced turbofan inlet and a separate flow exhaust system. Since the test vehicle includes no provisions for a separate core flowstream, the primary or core nozzle was truncated and replaced by the propulsion rig metering strut housing the powered drive. Thus, the inlet flow equals the fan nozzle exit flow, unlike a turbofan flight nacelle where the inlet flow splits into the fan and the core flow and exits separately from the two exhaust nozzles.

### 3.2.1. Inlet Aerodynamic Requirements

### Inlet Dimensions

Both inlet and exhaust systems are sized for maximum inlet corrected flow of 102.78 lb. At this flow rate, the inlet throat area is designed for maximum average Mach number at the throat  $(M_{th})$ ; to be equal to or below 0.75. At this flow condition, the fan operates at maximum specific flow of 43.5 lb/ft<sup>2</sup>. This value is consistent with current Allison Engine Company fan design criteria.







Figure 32. Swept vane design point Mach number distributions --- near-tip section.





Relative Mach Number



TE96-1047







Figure 36. Swept/leaned vane design point flowfield - near-tip section.



Figure 37. Swept/leaned vane design point flowfield — midspan section.



TE96-1051

Figure 38. Swept/leaned vane design point flowfield — near-hub section.

### Low Speed Requirements

This inlet is required to operate at maximum takeoff flow without internal flow separation for up to 20 degrees angle of attack (AOA) and at free stream Mach numbers ranging from 0 to 0.25, which are typical of levels encountered during aircraft terminal operations. No external inlet separation requirements have been considered; however, it is presumed in case of engine out or shut down, the nacelle forebody cowl will not separate at climbing speeds with AOA below 15 degrees. No crosswind and ground operational requirements have been considered either. For simplicity, an axisymmetric nacelle design with zero inlet droop angle is assumed adequate for this application.

### High Speed Requirements

The design cruise Mach number will equal 0.80. At the design cruise Mach number the fore and aft nacelle cowl contours are designed for minimal spillage and wave drag. Normally the engine nacelle is designed to have minimal total drag for a range of cruise Mach numbers, since the corresponding aircraft may be required to operate at different altitudes and flight Mach numbers. Generally, it is desirable to have a nacelle design so its overall drag remains constant or close to the design goal for flight Mach numbers at least 5-10% above the design cruise Mach value. This upper limit of Mach number is called the drag divergence Mach number ( $M_{dd}$ ). For this design,  $M_{dd}$  is fixed at 0.86.

### Other Constraints (Geometrical)

The nacelle aft cowl is designed to match the NASA propulsion simulator ducted prop drive rig. This requirement essentially sizes the overall test model dimensions, establishes the fan cowl and the core cowl boattail angles, and also locates the truncation point of the core cowl near the simulator metric station. The nacelle internal flow lines are constrained by the Allison wide chord fan design with a tip diameter of 22.0 in.

Since the model inlet flow and the fan duct flow are the same, the fan nozzle exit area is also sized to pass the maximum inlet corrected flow. Compared to the corresponding flight worthy nacelle, the fan nozzle is slightly larger than a scaled-up realistic fan nozzle design. The fan nozzle discharge coefficient (Cd) is assumed to be 0.984 (same value was used in the corresponding engine cycle) for choked flow nozzle conditions. No additional fan duct pressure loss has been included.

Initially, two different inlet/nacelle designs were developed to evaluate and compare the overall nacelle size required to incorporate various noise suppression linings. Figure 39 compares the nacelle aerolines for these configurations; however, due to program time and funding limitations, a single design with a compact inlet and diffuser length having (L) inlet/Dff of 0.50 was selected as a baseline nacelle configuration. The selected design provides adequate surface area, or space, for advanced acoustic treatments both in the inlet/diffuser region and in the fan duct. The duct and cowl lengths are sufficient, when scaled to the reference engine size, to accommodate an advanced thrust reverser design. The geometrical characteristics of the baseline nacelle are presented in Figure 40. The extra-long fan duct provides enough space to conduct tests with alternate OGV strut designs involving a set of sweep angles and varied axial lengths between fan trailing edge (TE) and leading edge of the OGVs. The fan and the core cowl contours have been designed to meet the above requirements with the external boattail angles of 10.8 and 8.8 degrees, respectively, consistent with wing mounted nacelles configured for low boattail pressure drag and with reduced nacelle/wing interference drag.

## 3.2.2 Aeroline Development

The inlet/nacelle contours have been generated using an Allison proprietary geometry code. This enabled an efficient, smooth flow path to be generated for enveloping engine hard points as well as maintaining the desired geometrical characteristics. NASA provided the attachment hard points on the







simulator flow path to maintain surface continuity between nacelle aft fairings and the drive shaft. Analytical or empirical 1-D techniques were used to provide preliminary performance projections prior to conducting a detailed CFD flow analysis. Figure 40 illustrates the nacelle aerolines along with important dimensions.

### 3.2.3 CFD Analysis

Inlet flow-field predictions using PMARC, a panel method code, were obtained to confirm the aerodynamic characteristics of the nacelle design. Figure 41 presents the baseline nacelle configuration analyzed, showing surface panels. Three flight conditions were analyzed using PMARC. These conditions are critical to the inlet design for engine operability and maximum cruise operation, and are as follows:

(1)  $M_{inf} = 0.2$ , AOA - 20 deg, Wcorr = 102.78 lb (2)  $M_{inf} = 0.8$ , AOA = 0.0 deg, Wcorr = 102.78 lb (3)  $M_{inf} = 0.0$ . AOA = deg, Wcorr = 104.5 lb

The analysis was conducted at several other conditions to calibrate the flow solution and the aerodynamic load calculation methods. Since this nacelle design will only be tested at low-speed conditions, condition (1) was used for the detail inlet/nacelle analyses. Typical surface flow distributions for the above conditions are enclosed in Figures 41, 42, and 43. Boundary layer analysis (Figure 44) was conducted using PMARC pressure distributions to provide surface skin friction Cf distribution on the inlet and nacelle to verify a separation-free flow.

### 3.2.4 Aerodynamic Loads

The pressure distribution obtained from the PMARC analysis at angle of attack was integrated over the nacelle length to obtain the resultant load and moment on the static structure due to operation at this condition. The results show both magnitude and point of application (fan face = 0.0) (Table II). These results were combined with the standard aerodynamic loads generated on the vane airfoils from deswirling of the fan rotor exit flow to determine the structural integrity of the static structure.



Figure 41. PMARC panels.













		C_1 Patch area/SREF	0.0000 0.0122 0.0000 0.0223 0.0000 0.0577 0.0000 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152 0.0152		<u>MX</u> <u>Patch area - in.</u> <sup>2</sup>	0.0000 176.3459   0.0000 320.4652   0.0000 31.0043   0.0000 831.0043   0.0000 219.3218   0.0000 2764.5623   0.0000 1686.5090   0.0000 392.1487		<u>Patch area - in.</u> 2	176.3459 320.4652 831.0043 219.3218 2764.5623 1686.5090 392.1487
		C n	0.0000 0.0000 0.0003 0.0020 0.0025 0.0034 0.0034 0.00334 0.0003 0.00334 0.0003		MZ	0.5839 -0.1111 24.1452 178.6471 47.4809 304.9659 -30.1228	<u>nt (in.)</u>	<u>Z bar</u>	0.0012 0.00012 0.0000 0.00001 0.00001 0.00000
ngle of attack.	ents	Cm	0.0000 0.0000 0.0000 0.0000 0.0000 0.0000	<u>oments (lb)</u>	XW	0.0000 0.0001 0.0000 0.0002 -0.0005 -0.0001 0.0000	<u>ch patch segme</u>	bar	0108 0013 3845 0285 4242 0000
<u>at 15 degree a</u>	Patch coefficio	CY	-0.0002 0.0000 -0.0112 -0.0218 0.0146 0.0146	<u>h loads and mc</u>	FY	-1.0351 0.0000 -66.5297 -129.3578 -12.3626 86.3899 0.0000	centroid of eac	Я	
lynamic loads		CA	-0.0262 -0.1673 0.0315 -0.0721 -0.0084 0.0000	Patc	FA	-154.9907 -991.4127 186.6494 -427.0309 -42.9820 0.0000 -81.6807	<u>Calculation of</u>	bar	3826 0000 8761 .0723 .0723
<u>Nacelle aerod</u>		CN	00000 00000 00000 00000 00000 00000 0000		EN	-0.0001 0.0000 0.0001 0.0001 0.0001 0.0000 0.0000		×	α, ⊷, rỷ ο, hẳ t <u>†</u> ,
		Name	Spinner Patch %%2 - fanface Patch %%3 - inlet duct Patch %%4 - nose Patch %%6 - nacelle Patch %%7 - exit end		Name	Spinner Patch %%2 - fanface Patch %%3 - inlet duct Patch %%5 - nose Patch %%6 - plume Patch %%7 - exit end		Name	Spinner Patch %%2 - fanface Patch %%3 - inlet duct Patch %%4 - nose Patch %%6 - plume
		Patch			Patch	1234567		Patch	< < < < < < < < < < < < < < < < < <

Table II. odvnamic loads at <u>15 degree angle of attack</u>.

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### 4.0 STRUCTURAL DESIGN

# 4.1 ROTATING COMPONENTS

The fan rotor assembly is composed of three primary components, an integral bladed disk (blisk) consisting of 18 airfoils and a hub; a spinner; and a torque sleeve. The blisk and torque sleeve are assembled to form a bolted assembly. The blisk is positioned radially on the torque sleeve at a pilot surface and retained through a bolted flange arrangement. Torque is transferred between the two components through a single shear pin. The spinner is threaded onto the forward portion of the torque sleeve to remove the need for attachment bolts and the associated access holes, which have produced additional tones in previous NASA test programs. The torque sleeve attaches to the drive rig through a force balance. Assembly is by way of four cross keys that are integral to the torque sleeve and mate with matching slots in the force balance. A titanium alloy, AMS 4928 in the solution treated and annealed state, was selected for the blisk and spinner due to its high strength to density ratio. Stainless steel, AMS 5659, was selected for the torque sleeve to meet the strength and life requirements of the cross keys.

# 4.1.1 Stress and Deflection Analysis

Structural assessment criteria employed to evaluate the structural integrity of the rotating components followed standard Allison practice for nonflight applications. Specific areas evaluated included rupture (tensile failure) speeds for both the blade and disk, section average and local tensile yielding, creep, low cycle fatigue life, and deflection under combined aerodynamic and centrifugal loading. No analysis of bird ingestion damage was attempted, but fan blade geometric parameters (such as leading edge radius) were constrained to lie within current engine experience.

All analysis was performed using the finite element method. A model of the blisk, torque sleeve, and spinner was generated for execution in the Allison proprietary finite element model (FEM) procedure, STRATA. Following Allison standard procedure, the analysis was conducted in two parts. A 2-D axi-symmetric analysis was performed on the disk, with the blade centrifugal loading applied as distributed tractions along the rim surface. The blade stresses were determined separately, with the airfoils represented by a mesh of 8-node meanline shell elements. The airfoil is attached rigidly to a plate oriented at the flow-path convergence angle. Based on Allison experience, stress concentration effects in the fillet regions are not modeled directly. Instead, a stress concentration factor ( $k_t$ ) is applied to the analytical results in the row of nodes immediately outboard of the hub boundary nodes. Standard values for  $k_t$  have been determined that yield an acceptable safety factor.

The structural audit sheets presented in Tables II and III summarize the results of this analysis as compared to material limits. Material properties contained in these tables were obtained from an Allison proprietary data base and include a sufficient sample size to establish statistical variations. For design assessment, the material properties used are those corresponding to three standard deviations (- $3\sigma$ ) below the mean of the material data base. Figure 45 presents the material properties of the titanium alloy used in the blisk and spinner, while Figure 46 presents similar data for the stainless steel used for the torque sleeve.

Airfoil, disk, spinner, and torque sleeve stresses were calculated at the design speed (N<sub>d</sub>) of 10,400 rpm, including the appropriate aerodynamic loads. Complete results of the stress analysis, in the form of isostress contour plots, is presented in Appendix E. The results presented for the airfoil include the effects of offsetting the stacking axis axially and circumferentially to balance the loading across the hub cross section. To ensure structural integrity, stress levels averaged over an appropriate section were required to be less than 0.8 of the tensile yield strength. For the disk, averages were obtained for both radial and tangential stress in the web and radial stress only around the flange attachment holes. For the airfoil, an average radial stress across the hub cross section was obtained. Results of the analysis indicate the maximum average stress occurs in the airfoil hub. The predicted levels are very low compared to the

	Common		Max stress	Max stress					
	Satisfy	Yes	Yes	Yes	Yes	Yes	Yes	Yes	Yes
	Location	Pressure surface	Pressure surface	Pressure surface	qnH	Leading edge	Pressuro surface	qлН	Leading edge
-	Calculated result	32.75 ksi	32.75 <b>ks</b> i	32.75 ksi	>3x106 cycle	>3 x 10 <sup>6</sup> cycle	32.75 ksi	27.22 ksi	19.48 ksi
	<u>-Ns</u>	112 ksi	3 70.36 ksi	>100.0 ksi	1000 cyc	1000 cyc	67.75 ksi	54.12 ksi	33.67 ksi
	21 20	e		e			т	ო	ς Ω
Allowoll	mean	124	77.32	>100.0			79.21	68.50	52.44
E	Parameter	Stress	Stress	Stress	Life	Life	Stress	Stress	Stress
Maximurr temp -	뛰	150	150	150	150	150	150	150	150
-	<u>Material</u>	Ti 6-4 STAN	Ti 6-4 STAN	Ti 6-4 STAN	TI 6-4 STAN	Ti 6-4 STAN	Ti 6-4 STAN	Ti 6-4 STAN	Ti 6-4 STAN
	<u>Design criteria</u>	0.8 Fty	1.0 Ftu $\left(\frac{Nmss}{Nb}\right)^2 \left(\frac{Nd}{Nmss}\right)^2$	1.0 Fcreep for 0.1% creep in 1000 test cycle life	1000 cycle test cycle life (0-maximum-0 cycles)	1000 cycle test cycle life (0-maximum-0 cycles)	>±15 ksi vibratory apapility, 3 x 10 <sup>7</sup> cyc at Nmss, maximum temperature	>±15 ksi vibratory apability, 3 x 10 <sup>7</sup> cyc at Nmss, maximum temperature	>±15 ksl vibratory apability, 3 x 10 <sup>7</sup> cyc at Nmss, maximum temperature
Calculation	method	FEM 3-D mean line shell	FEM 3-D mean line shell	FEM 3-D mean line shell	FEM 3-D Mean line shell	FEM 3-D Mean line shell	FEM 3-D Maan lina shell	FEM 3-D Mean line shelf	FEM 3-D Mean line ci shell
Critical	parameter	Average section stress	Average section stress	Average section stress	Steady state Kt = 1.4	Steady state Kt = 2.0	Steady state Kt = 1.0	Steady state Kt = 1.4	Steady state I Kt = 2.0
Objective/	UIANIAA	Tensile limit	Burst	Creep	LCF		НСЕ		-

# Table IV. <u>NASA 22-in. fan rig structural audit checklist — fan disk</u>.

				Ÿ	aximum		Attowable		•	Calculated		Satisfy	
Objective/	Critical	Calculation	Docido criteria	Material	temp. °F	Parameter	mean	Z	<u>sN-</u>	result	<u>Location</u>	<u>criteria</u>	Comments
concern	<u>parameter</u>	method	Design currents		1			4		0 00 hei	Weh	Yes	
Tensile limit	Average web rad	FEM axisymmetric	0.8 Fty	Ti 6-4 STAN	150	Stress at Nd	136.3	თ	122.8 KSI	184 06.7			
	stress		0 8 Ftv	Ti 6-4	150	Stress	123.9	e	111.6 ksi	12.41 ksi	Web	Yes	
	Average web tang			STAN		at Nd							
	stress				1	00040	103 0	e.	111.6 ksi	<1.0 ksi	Hole	Yes	
	Average hole rad		0.8 Fty	Ti 6-4 STAN	ner	at Nd	2	)	- - - -				
	stress					ġ	10 11	c	66 B ksi	2.93 ksi	Web	Yes	
Tensile	Average web rad	FEM axisymmetric	0.95 Ftu $\left(\frac{Nmss}{Nb}\right)^2 \left(\frac{Nd}{Nmss}\right)^3$	e Ti 6-4 STAN	150	at Nd	0.01	>					
	stress	,		i		Change	73.5	Ċ	66.8 ksi	12.41 ksi	Web	Yes	
	Average web tang		0.95 Ftu $\left(\frac{Nmss}{Nb}\right)^2 \left(\frac{Nd}{Nmss}\right)^2$	2 Ti 6-4 STAN	ner	at Nd	2	•			-		
	stress					č	70 5	۲	66 8 ksi	<1.0 ksi	Hole	Yes	
	Average hole rad		0.95 Ftu $\left(\frac{Nmss}{Nb}\right)^2 \left(\frac{Nd}{Nmss}\right)$	2 Ti 6-4 STAN	150	at Nd	0.07	0					
	stress					Ċ	70 5	c	66 8 ksi	11.85 ksi	Average	Yes	
Burst	Average section	FEM axisymmetric	0.95 Ftu $\left(\frac{Nmss}{Nb}\right)^2 \left(\frac{Nd}{Nmss}\right)^2$	2 Ti 6-4 STAN	150	stress at Nd	0.07	o			,		
	tang stres	S			ļ	Ċ		~	~100.0 ksi	11.85 ksi	Average	Yes	
Creep	Average	FEM axisvmmetric	1.0 Fcreep for 0.2% creep in 1000 test cycle	Ti 6-4 STAN	150	Stress at Nd		5					
	tang stres	Ś	life	· i		- 1		¢.	1000 cvc	-107 CVC	Fillet	Yes	
LCF	Peak	FEM axisvmmetric	1000 test cycle life (0-max-0 cycles)	Ti 6-4 STAN	nel			)				, , ,	
				Ti 6-4	150	Life		ი	1000 cyc	> 10 <sup>7</sup> cyc	HOIE	6D 1	
	910H			STAN									



Figure 45. Material properties of Ti6-4 (AMS 4928).

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Figure 46. Material properties of 17-4PH (AMS 5643): H 1100 annealed.

allowable values for the materials selected, easily satisfying the criteria. Since the blisk and torque sleeve form a bolted assembly, the integrity of the assembly must also be ensured. An axial stress field exists across the blisk to torque sleeve flange that tends to open this joint. Flange fastener sizes and assembly torque levels were determined based on the predicted axial stress levels across the flange to ensure separation will not occur. Torque transfer between the blisk and torque sleeve is accomplished through a dowel pin. The cross section of this pin was sized to carry the full rotor torque load at the maximum steady-state operating conditions in shear without help from the flange bolts.

The burst speed corresponds to the rotational speed at which either the airfoil or disk cross section is no longer able to support the centrifugal loading. Standard Allison design practice requires the burst speed be at least 25% above the maximum steady-state operating speed of the part. For this rig, the maximum operating speed has been defined as 105% of the design mechanical speed, or 10,920 rpm. Allison design criteria are intended to ensure tensile failure will occur first in the airfoil. For gas turbine disks with cross sections whose thickness varies radially, failure can occur as a result of either radial or tangential overload. For an ideally ductile material, redistribution of the cross-sectional loading would occur, delaying failure until the full cross section reached the material ultimate strength. As a result, the primary variable used in assessing disk tensile failure margin is the average stress across the full disk cross section. In certain cases the material may not be sufficiently ductile to fully redistribute the loading, resulting in failure due to overstress of a local cross section. To ensure a local failure condition would not affect the burst margin, average tangential and radial stresses over the disk web and average radial stresses around the flange holes were also determined Referring to Table III, the limiting tensile loading in the disk for this design is the result of tangential stress. Little difference is observed between averaging over the full cross section or the web cross section. The predicted levels for the disk are substantially less than the 0.95 of tensile ultimate allowed by the criteria at 125% of maximum steady-state operating speed. The maximum average stress levels again occur in the airfoil hub. Referring to Table III, the design criteria require these average levels to be less than the tensile ultimate for the blade material at 125% of the maximum speed. The predicted levels satisfy these criteria. Ratioing the airfoil average stresses by the square of rotational speed, burst is calculated to occur at a speed corresponding to 183% of the maximum steady-state operating speed.

Due to the limited running requirements for the rig, a minimum acceptable low cycle fatigue life of 1000 type 1 cycles (idle-maximum-idle) was established. The low cycle fatigue strength for AMS 4928 and AMS 5659 is shown in Figures 47 and 48 as a plot of cycles to crack initiation as a function of von Mises equivalent stress. For the airfoil, the life critical locations are in the hub fillet and along the leading edge. Stresses in the hub fillet were again determined through the application of a stress concentration factor of 1.4 to the finite element results, rather than through direct calculation. Along the leading edge the effects of small body foreign object damage have been included through the application of a stress concentration factor of 2. Based on these equivalent stress levels, minimum fatigue life in excess of 1 million cycles can be expected.

In addition to the stress results presented above, deflections were obtained from the finite element analysis. The predicted deflections in critical areas are shown in Figure 49. At the tip, the leading edge radial deflection of 0.020 in. was used to set the static clearance between the outer flow-path wall and the airfoil to preclude rubbing over the test speed range. At the pilot surface between the blisk and torque sleeve, the blisk was predicted to grow an additional 0.001 in. compared to the torque sleeve, due to the difference in elastic modulus of the two materials employed. In order to ensure accurate centering of the blisk on the torque sleeve at speed, this differential growth must not be allowed to open the pilot. To accomplish this, the mating pilot surfaces have been dimensioned to provide an interference fit at assembly. The predicted deflections were also used in an iterative procedure to determine the correct manufacturing coordinates to provide the intended aerodynamic shape at the design speed. The coordinates for the airfoil at static, 85% N<sub>d</sub>, and N<sub>d</sub> are tabulated in Appendix B.






Figure 48. Low cycle fatigue strength of AMS 5659 (17-4PH) at 70°F.

#### 4.1.2 Vibration Analysis

Vibration analysis of the integral bladed disk was carried out to define potential areas of vibratory response and to ensure adequate high cycle fatigue strength was available to allow operation over the entire design speed range. Specific consideration was given to avoidance of flutter over the rig operational envelope, placement of potential resonant conditions in speed ranges away from where substantial test time was to be accumulated, and satisfaction of minimum fatigue strength requirements over the entire bladed disk

Natural frequencies of the bladed disk system were obtained from finite element analysis at a series of rotational speeds. The finite element model consisted of a single airfoil supported on a pie-shaped sector of the disk. The periodic structure of the system was retained through application of cyclic symmetry boundary conditions along the edges of the disk sector. The airfoil was represented by a mesh of 8-node meanline shell elements, while the disk was modeled with 20-node solid elements. For completeness, comparisons of natural frequency and mode shape were made between the full bladed disk model and a cantilevered airfoil model. This comparison indicated insignificant levels of disk participation in the vibratory modes.

The results of the natural frequency analysis, in the form of a Campbell diagram, are presented in Figure 50. Plots of the deflected mode shape and resulting vibratory stress distribution are found in Appendix F. The diagonal engine order lines represent the locus of excitation frequencies produced by flow asymmetry with wavelength corresponding to the order number. Low order excitation (i.e., 2, 3, or 4EO) is typically the result of inflow total pressure or temperature distortion. Allison development experience indicates the coincidence of the fundamental bending (1B) and torsion (1T) natural frequencies with second and third-engine order should be avoided in speed ranges where significant operational time will be



Figure 49. Predicted rotor radial deflections.



Figure 50. Campbell diagram of blade.

accumulated. Of particular concern for the current design are the speeds corresponding to full power takeoff(8,840 rpm), approach (5,200 rpm), and design point (10,403 rpm) where the majority of the data is to be obtained. In recognition of this, the natural frequencies of the 1B and 1T modes were adjusted to provide a minimum 15% speed margin relative to 2 and 3EO at these critical speeds. Of secondary concern was excitation of higher order modes by the vane leading edge pressure field. There are 42 vanes in this stage, resulting in potential excitation at 42EO and its harmonics. It was not possible to provide a 15% speed margin between all natural modes and 42EO at the speeds of primary interest. Accurate prediction of the resonant response of a mode to excitation has not been achieved yet, thus precluding identification of specific modes whose resonant amplitudes will be unacceptably large. The design strategy was thus to minimize the number of modes experiencing resonant coincidence near the three speeds of except the 21st mode. Due to the relatively generous spacing between the rotor trailing edge and the stator leading edge, a weak excitation should be present, resulting in a low level response. Based on rig testing of similar components with similar rotor-to-stator spacing, responses of less than 10 ksi are anticipated.

A second area of major concern is the avoidance of flutter throughout the operational range of the rig. A combination of analytical and empirical methods have been developed at Allison for prediction of flutter onset. An analytical method, which predicts the aerodynamic damping associated with a specific modal deflection pattern, is available and has proven highly reliable. However, due to the method's mathematical formulation, it is only applicable in supersonic flows. The present design tip speed results in inlet relative Mach numbers too low for application of the analysis. To augment the analytical method, the product of chord\*frequency/(2\* inlet relative velocity). Empirical limits (minimum values) have been established at 0.2 for the fundamental bending mode and 0.6 for the first mode with significant torsional motion. For the current design, the calculated reduced frequencies of the relevant modes are 0.29 and 0.72. These satisfy the criteria.

Since total avoidance of vibration is seldom feasible, it is necessary to ensure typical levels of vibratory response will not result in fatigue failures. The endurance strength is the vibratory stress level in fully reversed bending that can be imposed on a material without producing high cycle fatigue failures. The endurance strength is reduced when a mean stress field is present, with the endurance strength approaching zero as the mean stress approaches the tensile ultimate. This material behavior is typically presented graphically in the Goodman diagram. In order to ensure a reasonable vibratory response will not result in fatigue data for notched specimens with theoretical stress concentrat ons,  $k_t$ , of 1.4 and 2.0 are used in the hub and edge regions respectively to account for fillet effects and for sign object damage. In other regions, fatigue data are based on unnotched specimens,  $k_t = 1.0$ . The fan des gn possesses a minimum fatigue allowable stress of 16 ksi in the leading edge region, which satisfies the criteria, Figure 51.

### **4.2 STATIC COMPONENTS**

The rig static structure is composed of a primary structural backbone connecting to the drive rig static force balance, a vane assembly composed of seven segments with six airfoils in each segment, and a series of spool segments forming the internal flow-path and nacelle outer profile. In order to isolate the acoustic effects of vane geometry, no separate structural frame is provided, forcing the vanes to become a load carrying member. The vane segments are tied to the static structure support by three bolts and a 0.250-in. shear pin. The shear pin provides the primary load path to ground, while the radial fasteners seat the vane segment against the static support. As discussed in Section 2.0, tour vane configurations are to be tested. In order to accomplish configurational changes with a minimum effort, all spool pieces downstream of the vane trailing edge are split axially to form bolted assemblies. The static structure support and vane assemblies are constructed of stainless steel, AMS 5643 (17-4 PH) heat treated to the H1100



specification, to provide the required strength and rigidity. The flow-path spool pieces are constructed of aluminum alloy, AMS 4127 (6061) in the T6 condition, to minimize overhung weight. Weight reduction was a priority to minimize the 1g deflection at the blade track and to facilitate handling during assembly. Additional outer flow-path pieces have been designed to adapt the rig to an existing bellmouth and variable area nozzle, allowing stage performance measurements to be acquired. To deal with the additional deflection resulting from the insertion of these pieces, provisions for external support have been provided.

# 4.2.1 Stress and Deflection Analysis

Structural analysis was carried out for each of the vane configurations at two loading conditions. The first loading condition represents standard rig operation and consists of the nacelle weight and aerodynamic loading generated on the vanes as they deswirl the rotor discharge flow. In the second condition, additional aerodynamic loads are applied as a result of operating the nacelle at an angle of attack to the wind tunnel flow. Structural assessment criteria employed to evaluate the integrity of the static components followed standard Allison practice for nonflight applications. Specific consideration was given to tensile rupture, tensile yielding, creep, low cycle fatigue, and deflection resulting from nonaxisymmetric loading. Due to the limited life, research nature of the rig, no provisions for containment in the event of an airfoil failure were included in the design of the nacelle. For this reason, human proximity to the rig during operation should be avoided.

All structural analysis was performed using the finite element method. A model composed of a 1/42 sector of the entire static structure, corresponding to a single vane passage, was generated for each of the four vane configurations for analysis in the Allison proprietary FEM procedure, STRATA. The vane inner band was descretized using 20-node solid elements. Beam elements were employed to represent the inner band attachment bolts, inner band shear pin, and the attachment bolts in the outer flanges. The rest of the structure was modeled with 8-node meanline shell elements. The static structure attachment to ground was through two spring elements at the pilot surfaces representing the rig static balance stiffness.

As previously mentioned, structural analysis of the rig was based on two loading conditions. Operation of the rig at an angle to the tunnel flow produces a nonaxisymmetric loading on the nacelle. Harmonic loading of the sector model was used to account for this asymmetry. As a result of the asymmetric load application, the stress and deflection patterns are also asymmetric. For nonsymmetric loading conditions, structural criteria are assessed at the worst location in the assembly. At the edges of the modeled sector, cyclic symmetric boundary conditions consistent with a split hoop are applied along the faces of the inner and outer bands. A secondary result of applying cyclic symmetry over a single vane passage width is that the model represents a structure with one bolt and one shear pin for each airfoil. This modeling inaccuracy will not affect the stress and deflection field away from the attachment points and was used to reduce computer resource requirements. To assess the stresses in the shear pin and attachment bolts, it was assumed removal of the additional constraints results in an equal increase in load in the remaining members. This produces a factor of six increase in the section stresses in the shear pin and a factor of two increase in the bolt stresses. This approach is not entirely accurate, but the resulting stresses are so low that a more accurate approach was deemed unnecessary. The bolted flanges on the outer duct pieces away from the vanes were not represented in detail in the finite element model. Bending stresses in the flanges were determined by hand calculation. A conservative approach was taken, requiring a single flange segment between two bolts to carry the entire nacelle bending moment due to angle of attack operation.

The structural audit sheet (Table V) summarizes the results of the analysis relative to the design criteria. The primary structural concern for the static components is the cocurrence of section yielding. Yielding is assessed using equivalent stress as defined by the Von Mises criteria. Referring to Table V, the peak equivalent stress occurs in the baseline vane hub trailing edge fillet when this vane is installed in the aft position, Figure 1b. This stress is 41% of the material yield, which satisfies the Allison criteria. As axial sweep is introduced, the peak stress levels decrease. This is a result of changes in the load transfer mechanism between the configurations. For the baseline vane, which has a radial stack axis, the nacelle loading is reacted out by the vane in pure bending about an axis normal to the airfoil plan view. This results in the majority of the load being transferred along the leading and trailing edge. As sweep is introduced, a portion of the nacelle load is transferred as tension parallel to the stack axis, similar to diagonal members in a truss. Since the section structural efficiency in tensile loading is greater than for bending, the resulting peak stress is reduced.

Table VI shows the circumferential variation in peak stress due to the load asymmetry for each of the vane configurations. Also shown in the table is the maximum stress due to the normal aerodynamic deswirl loads. Complete results of the stress analysis, in the form of isostress contour plots, is presented in Appendix G. Referring again to the audit sheet, the maximum stress in any of the flanges is found to be 7.5 ksi. These flanges are retained with 34 fasteners with 0.190-in. diameter. Standard torque levels for these fasteners will be sufficient to prevent opening of the flanges. The stress levels shown for the fasteners on the vane inner band reflect the Allison design practice of preloading fasteners at bolted joints to 80% of the material yield. In this application, the fastener stress is composed of 57 ksi due to preload and a 23 ksi bending stress from the vane loading. As in the rotating components, the design goal for low cycle fatigue life was 1000 type 1 cycles (minimum). Crack initiation is governed by local stress peaks; thus, the vane hub trailing edge fillet stress of 56 ksi will set the life potential for the static structure. The vanes are constructed from wrought 17-4PH stainless steel. Since the limiting stress occurs along an edge, a theoretical stress concentration of 3 is applied for life assessment to account for possible small object foreign object damage in this area. Based on these assumptions, the predicted low cycle fatigue life is 66,000 cycles.

In addition to the stress field induced in the vane and nacelle structure, operation at angle of attack will produce a deflection of the casing relative to the blade tip. The design is intended to have a uniform running clearance of 0.020 in. at the design rotational speed. The casing deflection at the blade track due to the nacelle loads is tabulated for the various vane configurations in Table VII. A maximum radial deflection of 0.006 in. is predicted and will occur in the swept and leaned configuration. Complete plotted results of the deflection analysis for both load conditions are presented in Appendix H.

### 4.2.2 Vibration Analysis

Vibration analysis of the static structure was carried out to define potential areas of vibratory response and ensure adequate high cycle fatigue strength was available to allow operation over the entire design speed range. Specific consideration was given to avoidance of flutter over the rig operational envelope, placement of potential resonant conditions in speed ranges away from critical test speeds, and satisfaction of minimum fatigue strength requirements over the entire structure.

Natural frequencies of the static structure assembly were obtained from finite element analysis. A finite element model of a 1/42 sector of the structure was generated for each of the four vane configurations and for both the flight and performance measurement ducting arrangements. Above the third natural mode, deflections tend to isolate in the vane assembly. To reduce the computational requirements, a reduced order finite element model representing the airfoil and vane outer and inner band was constructed to obtain these higher modes. Comparisons of the full system and reduced order models for a limited number of modes substantiated the accuracy of this approach. In the performance configuration, weight isolation for the inlet bellmouth and variable area nozzle will be provided. Based on the methods under consideration for providing this weight isolation, it was assumed they would not contribute to the system stiffness. The connection between Allison's static structure and the rig static force balance was simulated by NASA. Natural frequencies and mode shapes were calculated using the Allison finite element code, STRATA. Calculations of system response to unbalance were carried out using the Allison forced response code MODLRESP, running as a post-processor to STRATA.

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Objective/ concern	Critical parameter	Calculation <u>method</u>	Design criteria	N <u>Material</u>	Aaximum temp - ≜	Parameter	3σ <u>allowabie</u>	Calculated <u>result</u>	Location C	<u>configuration</u>	Loading condition	Satisfy <u>criteria</u>	Comments
Yield	Tensile + bending stress	3-D FEM	0.8 Fty	17-4 PH	120	Stress	111.2 ksi	56.7 ksi	Vane hub TE fillet	Aft vane I	Vane & AOA loads + weight	Yes	
	Tensile + bending stress	3-D FEM	0.8 Fty	A-286	120	Stress	80.2 ksi	80.2 ksi	Vane inner band bolt	Swept & leaned vane	Vane & AOA loads + weight	Yes	Assumes 57 ksi tensile stress due to bolt preload
	Bending stress	3-D FEM	0.8 Fty	A-286	120	Stress	80.2 ksi	19.2 ksi	Vane inner band shear pin	Aft vane	Vane & AOA loads + weight	Yes	
70	Bending stress	Hand calc.	0.8 Fty	17-4 PH	120	Stress	111.2 ksi	3.2 ksi	Vane outer band fwd flange	Aft vane	AOA loads	Yes	
	Bending ctreec	Hand calc.	0.8 Fty	Al 6061-T6	120	Stress	23.8 ksi	<1 ksi	Case/ vane outer band twd flange	Aft vane	AOA loads	Yes	
	Bending stress	Hand calc.	0.8 Fty	17-4 PH	120	Stress	111.2 ksi	7.5 ksi	Support E flange	Baseline vane	AOA loads	Yes	
	Shear stress	Hand calc.	0.55 Fty	A-286	120	Stress	55.1 ksi	<1 ksi	Static balance bolt	Aft vane	AOA loads	Yes	
LCF	Peak equiv stress + Kt	/ 3-D FEM	1000 cycles w/Kt = 3.0	17-4 PH	120	Cycles	1000	5300 cyc	Vane hub TE fillet	Aft vane	Vane & AOA loads + weight	Yes	
	Peak equiv stress + Kt	v 3-D FEM	1000 cycles w/Kt = 3.0	A-286	120	Cycles	1000	>10 <sup>5</sup> cycles	Vane inner band bolt	Swept & leaned vane	Vane & AOA loads + weight	Yes	
НСЕ	Peak equiv stress + K	v 3-D FEM + t Goodman diagram	>±15 ksi vibrtory capability w/Kt = 3.0 1 x 10 <sup>7</sup> cyc	17-4PH	120	Vibratory stress	>±15 ksi capability	18.6 ksi	Vane hub TE fillet	Aft vane	Vane & AOA loads + weight	Yes	

Table V. <u>Structural audit of static components.</u>

			Var	e	<b>D</b> 1/	Choonnin
Description	Tip LE	<u>Tip TE</u>	<u>Hub LE</u>	Hub TE	Bolt	<u>500 ar pili</u>
Maximum stress due to vane loads	9.1	<b>44</b> .0	36.3	48.3	10.0	10.0
Maximum stress due to vane + AOA load	ls + nacelle v	veight	50 F	20.0	12.8	18.6
Vane 1.90 deg	12.5	27.2	53.5	29.9	13.0	16.0
Vane 6 $\sim$ 135 deg	10.6	28.8	50.4	33.5	13.0	15.0
Vane 11 $\sim$ 180 deg	8.4	35.1	40.7	41.9	14.4	14.9
Vane 17, 225 deg	7.9	44.6	28.1	51.1	16.0	10.0
Vane 17, ~225 deg	8.0	48.8	23.2	54.2	16.6	17.4
Vane 22, 270 deg	7.9	46.6	25.9	51.8	16.4	15.0
Vane 27, ~315 deg	9.1	39.7	35.2	44.4	15.4	15.0
Vane 32, ~0 deg	11.9	30.9	48.3	33.8	14.4	18.0
Vane 38, ~45 deg						
			Va	ne	Rolt	Shear nin
Description	<u>Tip LE</u>	<u>Tip TE</u>	Hub LE	$\frac{\text{HUD IE}}{\sqrt{9}}$	17.8	14.4
Maximum stress due to vane loads	8.8	43.2	32.7	40.0	17.0	7.7.7
Max stress due to vane + AOA loads + r	nacelle weigh	t	40.7	20.2	16.6	19.2
Vane 1, 90 deg	12.4	25.8	48.6	27.2 21.9	16.0	16.8
Vane 6. ~135 deg	11.1	26.6	46.7	51.0	16.2	13.8
Vane 11 $\sim$ 180 deg	9.2	33.1	37.9	40.3	10.0	15.6
Vane 17, ~225 deg	7.7	44.0	24.7	52.0	10.0	17.0
Vane 17, 220 deg	7.1	49.2	18.9	56.7	18.4	17.4
Vane 27 ~315 deg	7.1	47.3	21.3	53.9	10.0	16.2
Vane 27, -010 deg	8.5	40.0	30.5	45.2	17.4	10.2
Vane $32$ , $-6$ deg	11.4	30.3	43.1	33.8	16.8	19.2
Valle 30, 43 deg			V			
			V	Hub TF	Bolt	Shear pin
Description	<u>Tip LE</u>	Tip IE	<u>HUD LE</u> 24.1	<u>1140 11</u> 14 2	11.8	15.6
Maximum stress due to vane loads	35.8	0.Z	24.1	14.2		
Maximum stress due to vane + AOA lo	ads + nacelle	weight	24 3	8.2	7.6	18.6
Vane 1, 90 deg	43.0	14.1	31.2	9.2	8.0	17.4
Vane 6, ~135 deg	38.5	12.0	21.2 25 0	15.3	9.6	16.8
Vane 11, ~180 deg	32.7	ð.3	10 0	24.2	13.2	18.0
Vane 17, -225 deg	29.2	3.6	10.4	27.2 76 7	15.4	18.6
Vane 22, 270 deg	29.5	3.6	16.4	20.2 วว จ	14.2	16.2
Vane 27, ~315 deg	32.0	3.7	19.2	22.0 112	11 A	14.4
Vane 32, $\sim 0 \deg$	36.7	7.1	25.3	14.0	0 Q 1 I .U	174
Vane 38, $-45 \deg$	42.5	12.3	32.5	9.2	0.0	17.7
			7	/ane		
			Hub I	E Hub TE	Bolt	Shear pi
Description	Tip LE	<u>110 I E</u> 11 5	7 7	4 46.5	14.4	16.
Maximum stress due to vane loads	32.8	11./ lo woight				
Maximum stress due to vane + AOA I	oads + nacell	e weigin	3 7	.2 33.1	3.4	18
Vane 1, 90 deg	24.0	Z.C	, , , ,	.0 37.0	4.6	5 16
Vane 6, ~135 deg	30.1	1	, , , , , , , , , , , , , , , , , , ,	4 43.8	11.2	2 17
Vane 11, ~180 deg	36.1	0.4	± J	<u>1 10.0</u>	20.4	<b>1</b> 6
Vane 17, ~225 deg	37.5	16.	0 14 0 17	2 10.8	22.9	- 3 18
Vane 22, 270 deg	34.5	17.	5 10 5 17	0 47A	18 2	s 15
Vane 27. ~315 deg	30.4	13.	5 12	**/.**	10.0	

Table VI. <u>NASA scaled fan rig nacelle vane static stress summaries (All values Von Mises equivalent stresses [ksi])</u>.

Tai	<u>b</u> le	VI	(cor	nt)
	_	_	_	

Description	TipIF	Tin TE	U.LTT	TT 1 mm		
Vane 32. ~0 deg	26.0		<u>FUD LE</u>	Hub IE	<u>Bolt</u>	<u>Shear pin</u>
Vana 29 AE da	20.0	5.6	3.6	42.3	10.4	16.2
valle 30, ~45 deg	22.3	2.1	5.5	35.1	3.0	18.6
						10.0

Notes:

All values are Von Mises equivalent stresses (ksi)

0 degrees is top dead center, with angle increasing counterc ockwise (aft looking forward)

Table VII.   NASA scaled fan rig nacelle blade track deflection summary (deflections in inches).							
Description Maximum deflection due to Vane 1, 90 deg Vane 6, ~135 deg Vane 11, ~180 deg Vane 17, ~225 deg Vane 22, 270 deg Vane 22, ~315 deg Vane 32, ~0 deg Vane 38,~45 deg	Radial vane + AOA 4.260e-03 3.350e-03 6.120e-04 -2.910e-03 -4.190e-03 -3.270e-03 -6.280e-04 2.880e-03	Baseline van Tangential loads + weight -7.030e-02 -7.310e-02 -7.470e-02 -7.360e-02 -7.080e-02 -6.880e-02 -6.780e-02 -6.850e-02	<u>Axial</u> -5.620e-03 -6.820e-03 -1.010e-02 -1.430e-02 -1.580e-02 -1.460e-02 -1.140e-02 -7.190e-03	Radial 4.940e-03 4.260e-03 1.220e-03 -3.410e-03 -5.290e-03 -4.100e-03 -8.100e-04 3.230e-03	<u>Aft Vane</u> <u>Tangential</u> -6.940e-02 -7.290e-02 -7.500e-02 -7.400e-02 -7.050e-02 -6.770e-02 -6.630e-02 -6.720e-02	<u>Axial</u> -5.660e-03 -6.570e-03 -9.510e-03 -1.350e-02 -1.500e-02 -1.410e-02 -1.110e-02 -7.240e-03	
<u>Description</u> Maximum deflection due to Vane 1, 90 deg Vane 6, ~135 deg Vane 11, ~180 deg Vane 17, ~225 deg Vane 22, 270 deg Vane 27, ~315 deg Vane 32, ~0 deg Vane 38, ~45 deg	Radial vane + AOA la 3.420e-03 3.170e-03 6.240e-04 -2.960e-03 -4.220e-03 -3.010e-03 -2.790e-04 2.820e-03	<u>Swept vane</u> <u>Tangential</u> oads + weight -5.820e-02 -6.100e-02 -6.250e-02 -6.140e-02 -5.850e-02 -5.690e-02 -5.690e-02 -5.690e-02	<u>Axial</u> 5.000e-03 3.340e-03 -7.120e-04 -5.620e-03 -7.200e-03 -5.540e-03 -1.530e-03 3.390e-03	<u>Swer</u> <u>Radial</u> 5.740e-03 4.850e-03 1.295e-03 -3.890e-03 -5.990e-03 -4.720e-03 -1.010e-03 3.730e-03	<u>ot and leaned</u> <u>Tangential</u> -1.710e-02 -2.120e-02 -2.371e-02 -2.240e-02 -1.860e-02 -1.610e-02 -1.460e-02 -1.500e-02	<u>Vane</u> <u>Axial</u> 1.340e-03 -1.100e-03 -6.668e-03 -1.320e-02 -1.510e-02 -1.270e-02 -7.160e-03 -6.390e-04	

The results of the natural frequency analysis of the full system, in the form of a Campbell diagram, are presented for the four vane configurations in Figures 52 through 55. Coincidence of the natural frequencies of these modes with first engine order (1EO) was of primary concern, since residual rotor unbalance would be capable of exciting a resonant response at such a coincidence. When configured with the flight inlet and nozzle, two modes were found that exhibited a 1EO coincidence in the steady-state speed range. It was not possible to adjust the frequencies of these modes sufficiently to move the resonant conditions outside the test speeds of the rig. Since both modes produced a result nt radial deflection at the blade track, excessive resonant amplitude could result in contact between the rotor tips and the casing. To determine the likelihood of such an event, a forced response analysis was conducted using a unit unbalance load applied in phase at the static structure support pilot surfaces. A damping of 6.3% (log decrement) was assumed, a conservative assumption based on Allison experience. The resulting blade track deflections for the pitch mode, which is the most sensitive to excitation, is presented in Figure 56.



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Figure 52. Campbell diagram for baseline vane in acoustic testing setup — assembly modes.



Figure 53. Campbell diagram for aft vane in acoustic testing setup — assembly modes.



Figure 54. Campbell diagram for swept vane in acoustic testing setup — assembly modes.



Figure 55. Campbell diagram for swept and leaned vane in acoust c testing setup — assembly mode.





Accounting for the 0.005 in. (worst case) of static deflection occuring during angle of attack operation, a minimum unbalance of 4 in.-Ib would be required to produce a rubbing condition for this mode. This level is two orders of magnitude larger than Allison balance requirements for hardware of this size. When configured in the performance mode, only one mode, labeled fore and aft in the Campbell diagram, coincides with 1EO within the steady speed range, Figures 57 through 60. A response calculation showed a residual unbalance greater than 10 in.-lb would be required to produce rubbing in this instance (Figure 61). The Campbell diagrams for the higher frequency modes, involving motion of only the vanes, are presented in Figures 62, 63, and 64 and correspond to the four test configurations. Since these modes involve vibration of only the vane segments, the results are independent of the nacelle configuration and do not change when the radially stacked airfoil is moved into the aft position. Since the rotor contains 18 blades, the primary concern for resonant vibration is the placement of the 18EO coincidences with the natural modes. Allison experience with fixed geometry vanes indicates resonant excitation of the fundamental bending, or 1B, mode should be avoided in the steady-state speed range. For all configurations, 1B-18EO resonance occurs well below the test speed range. This resonance should impose no restrictions on the test program. Three other modes are predicted to encounter resonant excitation within the steady-state speed range. The fundamental torsion (1T) and second bending (2B) modes exhibit a coincidence with 18EO at part speed conditions. For both of these modes at least a 15% speed margin exists between the resonant speed and the speeds at which the primary acoustic data will be acquired. Should an unexpectedly high response be observed in either of these modes, a modification to the test matrix to avoid the resonance can be implemented without compromising the test objectives. The second torsion (2T) mode of the vanes is also susceptible to an 18EO resonance. This resonance is predicted to occur approximately 5% below the design speed for the two swept configurations and at the design speed for the baseline configuration. Accurate prediction of aerodynamically induced resonant vibration levels remains beyond the state of the art. Review of recent Allison vane design experience reveals a number of successful core compressor stages have similar occurrences. In these stages, the measured response of the second torsion mode has been uniformly low. Since the present rig employs a much larger spacing between the rotor and stator than possible in a core stage, no unacceptable vibratory response of the 2T mode is expected and no attempt was made to change its natural frequency so as to avoid the 18EO resonance. Plots of the deflected mode shapes and resulting vibratory stress distributions are provided in Appendix I for the system modes and Appendix J for the vane modes.

While a relatively rare occurrence for a vane, avoidance of flutter throughout the operational range must be ensured. Allison has developed an empirical criterion for flutter avoidance based on reduced frequency as described in the rotating components section. Empirical limits for minimum acceptable values have been established at 0.2 for the fundamental bending mode and 0.6 for the fundamental torsion mode. The calculated reduced frequencies for the relevant modes for each of the vane configurations is presented in Table VIII. All configurations satisfy the requirements



Figure 57. Campbell diagram for baseline vane in performance calibration setup — assembly mode.



Figure 58. Campbell diagram for aft vane in performance calibration setup — assembly mode.



Figure 59. Campbell diagram for swept vane in performance calibration setup — assembly mode.



Figure 60. Campbell diagram for swept and leaned vane in perfor nance calibration setup — assembly mode.



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Figure 61. Blade track radial deflection versus fan unbalance — fore and aft mode of performance calibration setup.



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Figure 62. Campbell diagram for baseline and aft vanes — airfoil modes.



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Figure 63. Campbell diagram for swept vane — airfoil modes.



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Figure 64. Campbell diagram for swept and leaned vanes - airfoil modes.

A Goodman diagram for each of the four vane configurations is presented in Figures 65 through 68. As discussed in the rotating components section, Allison design criteria require the part be able to withstand a 15 ksi vibratory stress without experiencing a high cycle fatigue failure. This requirement must be satisfied at the location where the combination of mean stress and stress concentration effects ( $k_t$ ) is most restrictive. For all the vane configurations, the maximum mean stress occurs along an airfoil edge. At this location material data for a  $k_t$  of 3.0 is used to allow for the possibility of foreign object damage. All vane configurations satisfy the criteria.

		<u>Flutter</u>	Table ` parameter va	VIII. Ine configurations.		
Configuration	<u>1B - Hz</u>	<u>1T - Hz</u>	<u>75% chord</u>	<u>Velocity - ft/sec</u>	<u>1B (reduced)</u>	<u>1T (reduced)</u>
Baseline Aft vane Swept vane Swept and leaned	820 820 619 636	1320 1320 1362 1428	1.810 1.810 1.500 1.500	761 761 774 774	0.51 0.51 0.31 0.32 >0.2 required	0.82 0.82 0.69 0.72 >0.6 required



Figure 65. Goodman diagram for baseline vane.



Figure 67. Goodman diagram for swept vane.



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# APPENDIX A

# THE BASELINE LOW NOISE FAN: AERODYNAMIC DESIGN POINT BLADE AND VANE ELEMENT PERFORMANCE AND GEOMETRY OUTPUT

DATA 1	NO.L. 20111975915 219759197591	7,1975311975391. C 7,111111 7,1075311975391. C
NTERSTAGE AGE 1 OPY 1 OF	14.70 518.7 SPAN 10.00 200.00 500.00000000	PERCENT PERCENT 5 PAN 5
1 P 04/117 C	RESSURE EMPERATURE MER. VELOCITY 591:9 591:9 591:9 591:9 591:9 591:9 591:9 591:9 591:9 591:9	PRESSURE TEMPERATURE C MERTURE S821.8 5821.8 5821.8 5822.0 5925.0 5965.3 5966.7 5966.7
6:10:32	T01AL P T01AL T MACH N01E S446 S546 S546 S546 S546 S546 S546 S546	T01AL T01AL T01AL MACN S337 5337 5335 5335 5335 5335 5335 5335
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27 APR	ANNULUS ANNULUS CORRECTE CORRECTE 1.8869 1.8882 3.9888 3.9985 5.947 6.947 7.961 8.987 11.000	ANNULUS MASS FLC CORRECTI RADIUS INCHES 1.897 1.897 1.897 1.9925 5.959 6.959 6.959 8.976 8.976 8.976 8.976 9924

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27 API STATIOI ANNULU	MASS FI CORRECT RADIUS INCHESS 1.8834 2.9344 3.951 5.951 5.951 5.951 10.987 10.987 ANNULUS	MASS FL CORRECTION INCHES INCHES INCHES INCHES 10.9984 10.9988 10.9981 10.9921

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27 APR STATION STATION MASS FLO MASS FL

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27 APR 9 STATION N	ROTOR E MASS FLOW CORRECTED CORRECTED	TARADI US TRADI US 4.747 5.212 5.212 5.212 5.212 5.212 5.212 7.926 8.665 8.665 9.416 9.416 9.416 9.417 9.416 9.417 9.416 9.417 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.416 9.417 9.416 9.4176	RELAT INLET 1005 1005 1005 1005 1005 1005 1005 100	

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ıseline Low odynamic D	/ RATE/SQ.	701AL 701AL 17.53 17.53 17.53 17.53 17.53 18.29 18.29 19.56 20.24 21.21 21.21 21.21 21.21 21.21	LUS AREA. LUS AREA. TOTAL PRESSSURE 117.53 1
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94 NO. 9 EXIT 9	W RATE D FLOW RAT	AXIAL AXIAL Start S781.5 S79.5 S79.5 S79.5 S70.6 S70.5	FLOW RATE AXIAL AXIAL ELOCITY 580.25 580.33 580.33 580.33 580.33 580.33 586.33 562.11 552.11 552.11
27 APR STATION P ANNULUS	MASS FLOV CORRECTEL	RADIUS INCHES 1.CHES 5.9199 5.9149 6.568 6.568 7.986 7.986 7.986 10.1448 10.961 10.961	MASS FLOW MASS FLOW CORRECTED FNCHES 5.103 5.103 6.005 6.005 6.005 8.776 10.2493 10.2493 10.2493

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I Р 04/117 С	RESSURE EMPERATURE	V SELOCITY 5592556 5027.56 5027.56 5027.56 5027.59 6623.35 6633.35 6833.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 6835.55 75 75 75 75 75 75 75 75 75 75 75 75 7	EMPERATURE EMPERATURE MERCITY VELOCITY 5555.5 5563.5 5555.5 5563.5 55555.5 55555.5 5555.5 5555.5 5555.5 5555.5 5555.5 5555.5 5555.5 555
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г	MASS AVE. MASS AVE.	ABSOLUTE VELOCITY 7095.6 7095.0 7095.0 734.4 734.4 735.0 755.0 805.7 7885.7 7885.7 7885.7 7885.7	MASS AVE MASS AVE MASS AVE MASS AVE ABSOLUTE 675.9 675.9 681.6 681.6 681.6 681.6 734.1 734.1 734.1 734.1 734.1 734.1 734.1 735.5 812.5 815.1 815.1
* * * * * * *	TED) 1 SQ.IN	ATURES STATURES STATURES 505.8 505.3 511.9 511.9 511.9 5225.7 5225.7 5225.7 5228.8 5328.8	CTED) 2 50. IN 2 50. IN 2 510. 5 511.2 5 513.2 5 515.1 5 523.4 5 525.4 5 525.5 5 525.5
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94 NO. 11	EXIT 11 DW RATE	ED FLOW KA VELOCITY VELOCITY 5991.3 5991.3 5991.3 615.3 615.3 6615.3 6815.3 6815.3 6815.3 6815.3 6812.2 683.2	12 EU FLOW RATE AXIAL VELOCITY VELOCITY 5523 5571.0 5571.0 551.1 555.1 5
27 APR STATION	ANNULUS MASS FL	CORRECT RADIUS INCHES 5.0202 5.202 6.203 6.203 6.203 6.203 6.203 6.203 6.203 6.203 6.203 6.203 7.458 8.132 8.1323 8.1323 10.2323 10.964	MNULUS MASS FL MASS FL CORRECT RADIUS 5.159 5.159 5.159 5.159 5.159 5.159 5.220 8.220 8.887 9.887 10.257 10.257 10.257

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16:10:32	TOTAL PR	ABSOLUTE MACH NUJE 6100. 636 6655 7225 739 731
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27 APR 94 \*\*\*\* \* Baseline Low Noise Fan \*\*\*\*\* Explanation of \*\*\*\*\* Aerodynamic Design Point \*\*\*\*\* BLADE LOAD OUTPUT \*\*\*\* Aerodynamic Design Point \*\*\*\*\*\* POSITIVE AXIAL LOADS ON THE BLADES AND VANES CORRESPONDS TO POSITIVE TANGET WHICH IS OPPOSITE TO THE DIRECTION OF FLOW. POSITIVE TANGENTIAL LOADS INCLUDE A PRESSURE TERM AND A CHANGE ROTATION. THE BLADE LOADS INCLUDE A PRESSURE TERM AND A CHANGE IN MOMENTUM TERM AS DESCRIBED IN A TDR BY W.R.RATLIFF.3/30/61 ALL FORCES ARE IN POUNDS. NOTE THAT THE ACCOMPANYING PUNCHED CARDS ARE COMPATIBLE WITH NOTE THAT THE ACCOMPANYING PUNCHED CARDS ARE COMPATIBLE WITH

THE HUB AND TIP RAMP FORCES APPLY TO THE RAMP BETWEEN THE STATION THEY ARE PRINTED AT AND THE PRECEDING STATION EXCEPT STATION THEY ARE PRINTED. THE EXCEPTIONS WILL BE ASSOCIATED WHERE SPECIFICALLY NOTED. THE EXCEPTIONS WILL BE ASSOCIATED WITH THE SPLITTER LOCATION ON FAN COMPRESSONS. THE FIRST CALCULATING STATION AFTER A BLADE ROW WILL BE MARKED ROTOR. STATOR OR ICV WHICHEVER IS APPROPRIATE. THE EXIT OF THE BLADE STATOR OR ICV WHICHEVEN IN THE PRINTED CLEARANCE IN FRONT OF THE INDICATED STATION.

THE TOTAL AXIAL AND TANGENTIAL LOADS REFLECT THE TOTAL LOAD ON ALL BLADES OR VANES IN EACH ROW. THE TOTAL COMPRESSOR LOAD IS THE SUM OF ALL OF THE AXIAL LOADS ON ALL OF THE BLADE AND VANES IN THE COMPRESSOR. THE TOTAL HUB AND TIP RAMP FORCE IS VANES IN THE COMPRESSOR. THE TOTAL HUB AND TIP RAMP FORCE IS STATION TO THE LAST BLADE/VANE ROW EXIT(AS PER INITATION STATION). THE TATLE HADE/VANE ROW EXIT(AS PER INITATION STATION). THE TOTALS. NOTE THAT THE RAMP FORCES ARE USUALLY IN RAMP FORCE TOTALS. NOTE THAT THE RAMP FORCES ARE USUALLY BLADE AXIAL FORCES AND THE TOTAL HUB RAMP FORCE.

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DATA REDUCTION PAGE 4 COPY 1 OF 1	
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## APPENDIX B

## LOW NOISE FAN CONFIGURATION NO. 2: AERODYNAMIC DESIGN POINT BLADE AND VANE ELEMENT PERFORMANCE AND GEOMETRY OUTPUT

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95 NO. 9 EXIT 9	W RATE D FLOW RAT	AXIAL VELOCITY 588.3 586.3	588.2 596.0 607.2 630.6 638.5	641.5 630.6	FLOW RAT	ELAXIAL FELOCITY 599.8 599.8 6010.5 610.5 650.9 650	
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eline Low dynamic De	RATE/SQ. F US AREA 2	TOTAL PRESSURE 17.63 17.72 17.22 18.28	18.88 199.555 200.222 21.21 21.47 21.47	RATE/SQ. F US AREA 2	PRESSURE 117.63 117.63 117.63 118.28 288.28 20.22 20.22 20.22 21.21 40 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.21 21.22 21.22 21.21 2
Bas Aero	FLOW	RADIAL /ELOCITY 19.80 18.165 21.37	1222.52 18.728 9.94 4.83 493 493 493	FLOW	RADIAL FLOCTITY 159.557 221.888 221.888 227.117 227.112 12.501 133.552 13552 1355555 1355555555555555555555
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95 NO. 13 EXIT 13	OW RATE ED FLOW RA	AXIAL VELOCITY 613.4 611.5 610.8 616.8	649 649 649 695 695 695 695 695 695 695 695 695 69	14 OW RATE ED FLOW RA	V ELAXIAC 5884.14 5948.14 5024.51 5024.51 7062.41 7055.41 7052.41 7055
10 APR STATION ANNULUS	MASS FL CORRECT	RADTUS INCHES 5.166 5.792 6.792	6.878 7.510 8.170 8.847 9.538 10.241 10.241	ANNULUS MASS FL	RA RA RA RA RA RA RA RA RA RA RA RA RA R
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eline Low dynamic D	RATE/SQ. US AREA TOTAE/SQ. PRESSURE 17.72 117.72 117.72 118.88 188.88 188.88 188.88 187.20 20.22 20.27 21.47 21.47 21.47 21.47	RATE/SQ. LUS AREA TOTAL PRESSUR 17.53 17.53 17.53 17.53 17.53 17.53 17.53 19.55 20.72 21.21 21.47 21.21
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95/100 EESSURE MPERATURE VEMER: VEMER: VEMER: VEMER: S504:6 5535:2 5535:2 5534:8 734:8 734:8 736:1:1 756:3 756:2 756:2 756:2 756:2 756:2 758:4	RESSURE EMPERATURE MER: VELOCITY
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10 APR STATTON ANNULUS ANNULUS ANNULUS ASS FIC CORRECTI 5.968 5.968 5.903 6.303 5.968 5.903 6.303 5.968 7.7389 6.303 6.303 6.303 6.303 5.968 7.7389 6.303 6.303 6.303 5.968 7.7389 6.303 5.968 7.7389 6.303 5.9587 10.959 10.959	ANNULUS MASS CORRECT CORRECT CORRECT S S S S S S S S S S S S S S S S S S S

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	15:45:31 • TOTAL F	ABSOLUTE MACH NO.	4477 4477 4070 4070 4070 4070 4070 4070	682	TOTAL P	ABSOLUTE MACH N0. .410 .427 .452 .453	259 89 89 89 89 89 88 89 88 78 88 78 88 78 88 78 88 78 88 78 88 78 88 78 88 78 88 78 88 78 7
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v Noise Fan Jesign Point	FT, 40.52 1.96 SQ. FT	STATIC PRESSURE 15.25 15.25 15.25	1125.27 1125.27 125.27	15.40	FT. 40.52 1.96 SQ. FT	STATIC PRESSURE 15.28 15.29 15.29 15.29	155.30 155.31 155.31 155.31
tseline Low odynamic C	/ RATE/SQ.	TOTAL PRESSURE 17.33 17.59	18.03 18.69 20.60 20.60 21.98	21.02	LUS AREA.	PRESSURE 17.16 17.33 17.59 18.05	20.07 20.060 21.188 21.188 21.188
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DATA REDUCTION PAGE 2 15:45:31 95/100 COPY 1 OF 1 MAKE AVERAGED A/A*	FREE STREAM MIN PASSAUE 1.019 1.035 1.001 1.104	
* * * * * * * * *	MEANLINE Solidity 1.657	1.559
⊐e Low Noise Fan amic Design Point	STAGE	. 8956
* * * * Basel 11 * * * * * Aerodyn	* * * * MASS AVERAGED D-FACTOR	.4346 .3074 .3710
10 APR 95	AIRFOIL	ROTOR 1 STATOR 1 AVERAGE

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## APPENDIX C

## LOW NOISE FAN CONFIGURATION NO. 3: AERODYNAMIC DESIGN POINT BLADE AND VANE ELEMENT PERFORMANCE AND GEOMETRY OUTPUT

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		NN 2011111 201010100001	2001559107591. 219759107591.
NTERSTAGE AGE 1 OPY 1 OI	14.70 518.7	PERCENT SPAN 2000 2000 2000 2000 2000 2000 2000 20	900.00 900.01 900.01 900.02 900.02 900.01 900.00
95/192 C	KESSURE MPERATURE	VE MER. SSP1:99 SSP1:9	RESSURE EMPERATURE VENCTY VENCTY S882.5 5892.7 5882.5 5893.9 5993.9 5993.9 5993.9 5966.9 5066
9:16:10	TOTAL PH TOTAL TE	ABSOLUTE MACH NO MACH	. 101AL PL ABSOLUTE ABSOLUTE ABSOLUTE 553395 5550 5550 5550 5550 5550 5550 55
	MASS AVE. MASS AVE.	ABSOLUTE VELOCITY 591.9 595.9 505.9 505.0 505.0 505.0 505.0000000000	AAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAAA
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wept (Only) esign Point	FT. 39.18 2.62 SQ. FT	PRESSUR 12:00 10:00 10:00 10:00 10:00 10:00 10:00 10:00 10:0	FT: 39.18 2.62 SQ: FT STATIC PRESSURE 12:09 12:09 11:98 11:98 11:98 11:98 11:98
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95/192	RESSURE EMPERATURE	VELOCITY 5660.00 5880.0 5882.1 5882.1 5927.1	598.6 600.6 601.9 602.4	RESSURE EMPERATURE	VELOCITY S506.05 S706 S706.05
9:16:10	TOTAL P	ABSOLUTE MACH NOT 515 523 5330 5330 5330 5461 5461	· · · · · · · · · · · · · · · · · · ·	TOTAL P	ABSO LUT MACH MACH NACH AGU 100 100 100 100 100 100 100 100 100 100
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95 NO. 3 EXIT 3	W RATE D FLOW RAT	VELOCITY S568.0 5575.7 5875.7 5871.9 5821.9 5822.0 5925.7	600.6 601.9 602.4	4 W RATE D FLOW RAT	CE AXIAL CE AXIAL CE AXIAL CE AXIAL CE COCLIA SS002100 CE AXIAL CE AXIA
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95/192	RESSURE EMPERATURE MER. VELOCITY VELOCITY VELOCITY VELOCITY 8472.2 5441.3 5699.3 5699.3 5699.3 5692.1 6612.4 6612.4 6612.4 6612.4 6612.4 6612.4 6627.7 6628.8	PRESSURE PRESSURE C MERATURE V MER. V MER. V MER. V MER. S549.9 549.9 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 650.1 850.100.1 850.100.10
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11 JUL STATION	ANNUCUS MASS FL CORRECT INCHES INCHES 1.1229 6.1150 6.1150 6.1255 6.1255 6.1255 70056 879 979	ANNULUS MASS FL CORRECT RADIUS

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9:16:10	TOTAL PRI TOTAL TEN	ABSOLUTE MACH NO. 628 6332 642 655 655 655 663 663 663 663 663
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95 NO. 8	EXII I W RATE D FLOW RAT D TIP SPEE	KE KX IAC S586.8 5766.8 5766.8 5283.6 5283.6 5283.5 5285.5 5285.5 5285.5 5275.5	MACH EXH EXH EXH SS55 SS546 SS53 SS53 SS53 SS53 SS53 SS53 SS53 SS5	FFUSION ACTOSION ACTOSION 336 336 430 4430 4457 4451 4451 4451 4451 4452 4451
NOT TATE	MASS FLO CORRECTE	RADIUS RADIUS 4.CHUUS 5.208 65.208 65.208 7.0145 7.0145 7.0145 7.0119 7.128 7.	RE 11100555 11100555 11100555 11100555 11100555 11100555 1110055 110055 110	

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DATA 1.40000400000 NO. L. NO н INTERSTAGE PAGE 6 COPY 1 OF ЦO PERCENT S PAN S PAN 111.25 299.99 200.88 200.89 200.89 200.89 200.88 200.89 200.89 200.89 200.89 200.89 200.89 200.89 200.89 200.89 200.88 200.89 200.80 200.80 200.80 200.80 200.80 200.80 200.80 200.80 200 20.25 20.25 571.5 PRESSURE TEMPERATURE PRESSURE TEMPERATURE 95/192 ABSOLUTE MACH NO. 6657 6645 6645 6645 6673 6673 6677 677 ABSOLUTE MACH NO. .6679 .6551 .6551 .6551 .6566 .6666 .6833 .6583 .6579 .6579 9:16:10 TOTAL TOTAL AVE. AVE. ABSOLUTE VELOCITY 73355 719.2 717.2 718.9 718.9 758.5 768.5 768.5 772.1 772.1 772.1 772.1 777.1 777.1 MASS MASS MASS MASS ATV3 TEMPE RATURES 101AL STATIC 549.4 STATIC 559.8 S02.1 5554.2 S111.3 5555.1 S111.3 565.1 S111.3 565.1 S111.3 565.1 S111.3 565.1 S111.3 565.1 S12.3 563.0 S330.1 583.9 S330.1 (CORRECTED) = 310.8 SQ.IN 36.95 (CORRECTED) SQ. FT = 305.2 SQ.IN \* \* Swept (Only) Vane Design Point \* 36.28 ( SQ. FT PRESSATIC 132.95 133.95 133.95 144.15 144.15 155.55 155.55 16.07 PRESTIC PRESSUR 13.555 13.555 13.555 13.555 14.142 15.5588 15.5588 15.5588 15.5588 15.5588 15.5588 15.5588 FT. 2.12 FT. 2.16 T01AL PRESSURE 17.53 17.63 17.63 17.90 18.88 19.55 19.55 20.23 21.21 21.40 21.40 PRESSUR 17.53 17.53 17.53 17.53 17.53 18.88 20.23 20.23 20.23 21.21 21.21 21.40 21.40 FLOW RATE/SQ. ANNULUS AREA / RATE/SQ. V RADIAL V ELLOCITY 955.19 875.55 589.554 724.48 124.48 124.48 124.48 124.48 RADIAL ELOCITY 1241.777 1241.777 123.641 87.001 87.584 587.584 588.55 5887 5.897 5.897 FLOW 102.78 78.31 102.78 78.31 VERT VERT VERCUT Used to RATE RATE ە 6 VELOCITY VELOCITY 592.5 592.5 592.9 551.9 551.2 643.2 643.2 643.2 643.2 643.2 VE AXIAL 6000.17 5987.20 5987.8 5988.8 5643.44 5664.9 5664.9 569.9 569.9 RATE FLOW RATE FLOW 95 NO. EXIT MASS FLOW CORRECTED MASS FLOW CORRECTED 5 STATION STATION ANNULUS RADIUS INCHES 5.1655 5.1655 6.109 6.109 6.109 6.109 8.774 9.490 10.247 1 ANNULUS

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11 P1 95/192 C0	IMPERATURE MER. VELOCITY VELOCITY S895.8 5033.7 6538.9 6538.9 6538.9 6538.9 6538.9 6538.9 6538.9 6538.9 6538.9 6538.1 681.7	RESSURE MERATURE MERATURE MERATURE S833.1 5883.0 5883.0 5883.0 6683.5 6683.5 6683.5 6683.5 715.1 712.9		
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nfigure Aero	FLOW FLOW ANNUL ANNUL 775.755 775.755 775.755 775.755 735.255 552.2555 552.255 552.255 552.255 552.255 552.255 552.255 552.255	FLOW ANNU RADIAL VELOCITY 855.21 777.28 565.267 356.268 356.268 356.268 245.406 12.766 256.268 356.268 356.268 257.268 257.268 256.268 256.268 256.268 256.268 256.268 257.268 256.268		
Used to co + + + +	TE 102.78 WHIRL 78.31 VMIRL VELTY VELTY VELTY 102.78 380.99 371.7 371.7 371.7 371.7 371.7 371.7 371.7 371.7 371.7 371.7 373.5 412.7 412.7 412.7 393.3 393.3	TE 102.78 WHIRL 78.31 VELOCITY 78.31 386.1 386.1 3972.7 393.2 413.2 393.2 393.2		
95 NO. 11	EXIT 11 EXIT 11 ED FLOW RATE AXIAL VELOCITY 5881:57 5881:77 5881:77 5881:77 5881:77 5881:77 5881:77 5881:77 5881:77 6811:77 6811:77 6811:77 6811:77 6811:77 6811:77 6811:77 6811:77	12 ED FLOW RATE AXIAL VELOOW RA 576.8 598.1 598.8 598.8 598.8 598.8 598.8 598.8 598.8 598.8 598.3 598.3 598.3 598.3 598.3 598.3 504.5 712.9 712.9		
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	93/192 CUP RESSURE	VELOCITY F	500.5 609.5 655.0 679.5	715.4 725.7	1.62/	RESSURE EMPFRATURF	VELOCI S922.6 5922.6 631.0 631.0 631.0 8 8 7 8 8 7 8 8 7 8 8 8 8 8 8 8 8 8 8	0.000 8883 89045 80045 80045
01.31.0	TOTAL P	ABSOLUTE MACH NO.	. 666 . 666 . 715	.731 .741 .743	TC/.	TOTAL PI	ABSOLUTE MACH NO. 633 633 663 662 662 662 662 662 662	708 708 8008
	MASS AVE MASS AVE	ABSOLUTE VELOCITY 697.8 696.2	712.00 739.8 797.8	817.0 830.2 835.0	+ • • • • •	MASS AVE	ABSOLUTE VELOCITY 700.5 700.6 700.6 713.0 713.0 758.7 778.7	789.8 796.7 798.0 798.0
* * * *	CTED) .8 SO.IN	RATURES STATIC 507.9 508.8	5113.8 5113.8 515.7 517.7	5222 5222 524 524 524 524 50 524 50 52 50 50 50 50 50 50 50 50 50 50 50 50 50		TED) 9 SQ.IN	KATURES 507.5 508.4 509.9 511.8 511.8 512.1 520.3	1.400 8020 8020 8020 80 80 80 80 80 80 80 80 80 80 80 80 80
Vane t	(CORRE	TEMPE TOTAL 548.4 549.2	5705.11 5705.12 5705.12	582.8 582.8 583.6		(CORREC = 301.	TEMPE 5559.28 5559.28 5559.28 5559.28 5655.14 5565.14 5565.14 5565.14 5565.14 5565.14 5565.14 5565.14 5565.14 5559.28 5657.14 5559.28	5280 5280 5280 5280 5280 5280 5280 5280
Swept (Only) Design Point	FT. 38.25 2.05 SQ. F1	STATIC PRESSURE 13.48 13.57 13.57	11111 14,203 14,203 14,203	14. 73 14. 73 15. 00		FT: 37.35 2.10 5Q. FT	PRESSURE PRESSURE 13.45 13.65 13.65 13.65 13.65 14.07 14.07 14.07 14.07	15. 18 15. 37 15. 37
Axially odynamic (	RATE/SQ. LUS AREA	TOTAL PRESSURE 17.63 17.73 17.90	18.29 19.55 20.23	21.21		RATE/SQ. US AREA	T0TAL PRESSURE 17.63 17.93 17.93 18.29 18.88 198.55 20.23	211.21
onfigure	FLOW	RADIAL VELOCITY 75.93 76.44 75.34	72.43 607.38 20.86 20.36	16.40 16.40		FLOW	VELOCITY VELOCITY 76.42 79.35 882.35 883.33 875.53 875.555.53 875.555.555 875.5555.555555555555555555	-41.61
Used to + + + +	VTE 102.78	WHIRL VELOCITY 375.7 368.7 361.3	368.0 386.5 404.4 422.3	421.3 413.0 393.2		TE 102.78	VELOCITY VELOCITY 3555.7 3557.11 3562.11 3562.11 3562.11 3565.18 8107.65 4107.65	410.8 403.4 855.1
L 95 N NO. 13 5 EXIT 13	LOW RATE	AXIAL VELOCITY 583.1 585.6 591.3	627.2 627.2 677.6 698.3	714.8	14	OW RATE ED FLOW RA	VELACIAL VELACIAL 597.13 597.13 608.5 643.5 643.5 643.5 573.7	682.5 689.1 687.3
STATION STATION ANNULUS	MASS FI CORRECT	RADIUS INCHES 5.229 5.472 5.865	6.951 6.951 8.227 8.894	9.572 10.261 10.967	ANNULUS	MASS FL CORRECT	RADIUS INCHES 5.256 5.2556 5.256 5.256 5.255	9.816 10.505 11.199

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DATA 1		2001 2011 2012 2012 2012 2017 2017 2017	241414 241444 24238197591	
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95/192 0	PRESSURE FEMPERATURE EFFICIENCY	VELOCITY VELOCITY S334.66 S334.66 S334.66 S334.66 S345.00 S488.58 S488.58 S488.58 S488.58 S488.58 S488.58 S488.58 S488.56 S488.58 S488.56 S488.58 S488.56 S588.56 S5856 S588.5		
9:16:10	TOTAL PIABATIC	ABSOLUT MACH NO • 4467 • 4467 • 4856 • 5807 • 5808 • 5808 • 5808 • 5661 • 6661	LYTAGE FICTIENCY 881.0 885.0 992.8 992.8 993.1 991.9 93.1 993.1 87.3 87.3	
	MASS AVE MASS AVE STAGE AL	ABSOLUTE VELOCITY 5324.6 5345.0 570.7 570.7 570.7 5887.5 6887.5 6887.5 7188.6 776.9 776.9 796.7 796.9	69.444 69.66 69.66 69.66 69.66 69.66 69.74 69.74 69.74 69.74 69.74 69.74 69.74 69.74 69.74 69.74 69.74 69.74 60.74 70.74 7	•
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95 15 1	EXIT 1 W RATE D FLOW RATI	KAT10 EECOCITY 5520.8 5541.9 5688.5 667.0 667.0 718.5 718.5 718.5 718.5 718.5 718.5 725.5 725.5 725.5	MACH NOS. EXIT 05.1474 5474 54055540 5540 5540 5535 6661 6661	FFUSION ACTOR 357 357 357 357 3347 3347 3347 3326 3326 3326 3326 3326 3326 3326 332
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	95/192 RESSURE	EMPERATURE MER. VELOCITY 471.6	00000 0000 0000 0000 0000 0000 0000 0000	728.6 728.6 7902.1 804.8	RESSURE EMPERATURE	MELOCITY VELOCITY 445.09 467.2 544.7	603.1 759.8 799.8 798.8 798.8
01.31.0	OT ST TOT .	ABSOLUTE MACH NO. .431	450 526 570 570	. 700 . 700 . 713	TOTAL PI	ABSOLUTE MACH NO. .394 .413 .413 .441	. 535 535 6631 708 708
	MASS AVE	ABSOLUTE VELOCITY 471.6 486.3	507.4 544.4 643.8 693.8 693.8	728.6 762.1 790.4 804.8	MASS AVE. MASS AVE.	ABSOLUTE VELOCITY 445.9 467.2 497.7 544.7	6003.1 7109.8 749.5 798.8 798.8 798.8
* * * *	CTED) 7 SO IN	ATURES STATIC 529.9 529.5	529.0 530.0 530.0 530.0 530.0 530.0	531.3 530.8 530.8	TED) 2 SQ.IN	ATURES STATIC 531.9 530.1 529.5	0.011 8.788 200.11 8.7888 200.11 8.78888 200.11 8.7888 200.11 8.78888 200.11 8.788888 200.11 8.788888 200.11 8.78888 200.11 8.7888888 200.11 8.788888 200.11 8.7
Vane.	CORREC	TEMPE 548.4	5559.28 5559.28 570.14	575.5 579.7 582.8 583.9	(CORREC = 282.	TEMPER 5499.24 5550.82 5550.80	582950711 5882950711
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axially S odynamic D	LUS AREA.	PRESSURE	18.04 18.70 19.41	21.02 21.23 21.11	RATE/SQ. F LUS AREA 3	T0TAL PRESSURE 17.10 17.28 17.28 18.57 18.00 18.70	20.09 210.05 21.23 21.13 21.11
onfigure	FLOW	RADIAL VELOCITY 5.29 6.54	-11.26 -11.26	- 22.15	FLOW	KADLAL /ELOCITY -1.14 06 -78 82	81.00 81.00 81.00 80.10 80.00 80
Used to	E 102.78	WHIRL VELOCITY	00000	000	E 102.78 E 79.14 WUTDI		000000
95 No. 16 EXIT 16	W RATE D FLOW RAT	AXIAL VELOCITY 471.6 486.2 507.4	544 3 593 8 660 5 728 5 5 7	761.8 790.1 804.6	FLOW RAT	ELOCITY 465.29 467.2 544.7 544.7 503.1	700.8 710.8 710.8 709.8 708.8 708.0
STATION STATION ANNULUS	MASS FLO	RADIUS INCHES 5.683 5.938 6.340	6.840 7.391 8.545 9.134	9.732 10.340 10.967 ANNULUS 17	MASS FLOW CORRECTED RADTUS	1000 5 691 5 691 5 691 5 691 5 60 5 60 5 60 5 60 5 60 5 60 5 60 5 60	8.543 9.126 9.721 10.331 10.963

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NTERSTAGE 1 AGE 11 OPY 1 OF	20.03 571.5	PERCENT SPARC SPARC SPARC SPARC SPACENT SPACEN	20.03 571.03 571.03 11.04 11.04 11.05 573.04 222.97 22.24 56.25 57.00 99.25 99.20 99.20 99.20 99.22 99.22
1 95/192 0	RESSURE EMPERATURE	VELACCITY VELOCCITY 4455.5 4645.5 4665.5 6605.5 6605.5 7192.3 7926.5 7872.5 7872.5 7872.5	RESSURE FEMPERATURE MERTURE MERTURE MERTURE VELOCITY
9:16:10	TOTAL P	ABSOLUTE MACH NO. . 394 . 414 . 442 . 442 . 587 . 587 . 587 . 588 . 6632 . 688 . 688 . 688 . 692 . 688 . 692 . 688 . 692 . 602 . 702 . 702	. T0TAL F . T0TAL F ABSOLUTE MACH V01AL 4456 5440 5590 5590 5664 5664 5691 6691
	MASS AVE. MASS AVE.	ABSOLUTE 4650555 4645555 4645555 66055556 7749556 77495555 7955555 7955555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 795555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 7955555 79555555 79555555 795555555 795555555 795555555 7955555555	MASS AVE MASS AVE ABEOLUTE 4510.3 4721.8 5503.7 713.6 5503.1 713.6 5503.1 7248.0 7748.0 7744.0 7877.8 7877.8 78179
* * * * *	TED) 0 SO.IN	SATURES SATURES 5331:09 55289:33 55288:68 55288:68 55288:88 5328:58 5328 5328:58 5328 5328 5328 5328 5328 5328 5328 53	6 50. IN 847URES 5301.5 528.5 528.5 528.5 528.3 528.3 531.1 531.1
Vane,	CORREC	11111111111111111111111111111111111111	CCORE TEMPE 5599:281 5599:281 5599:282 5755:192 57555:192 57555:192 57555:192 57555:192 57555:192 57555:192 57555:192 57555:192 57555:192 575555:192 575555:192 5755555 57555555555555555555555555555
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SUMMARY 1 1 OF 1		<b>ພ</b> ແ ო
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9:16:10 95/192	MACH SPECIFIC NO. FLOW 1.144 43.13 35.58 .634 37.35 40.27	TOTAL INLET AXIAL HUB TIP MEAN 7.4 10.6 686.2 2.2 29.2 643.5
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\* \* \* \* \* Aerodynamic Design Point \* \* \* \* \* \* SARP CL/ SPAN 0000 LOSS MODIFIERS REYNOLDS REYN# CL-LOSS NUMBER MODIF COEFF. 1788482. 000 00000 732511. 000 ALLISON SURGE MARGIN FLOW LOADING ASPECT COEF PARM RATIO 1.064 .4867 1.754 1.064 .6188 2.456 11 JUL 95 ROTOR 1 VANE 1 AVERAGE AIRFOIL

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11 JU STATIOI ROTOR	MASS FI CORRECT	SL PER- NO CENT SPAN	10220.8 114 1008808 14 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8	10 22 23 0 20 24 40 20 34 40 2	TATOR N	MASS FL	C PER- 0 CENT SPAN		700 700 700 700 700 700 700 700 700 700
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* * *** MEANLINE SOLIDITY 1.556 1.597 1.597	
swept (only) Vane Design Point * STAGE REACTION .8945	
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11 JUL 95 AIRFOIL ROTOR 1 STATOR 1 AVERAGE	

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DATA REDUCTION PAGE Used to configure Axially Swept (Only) Vane ATV3 \* \* \* Aerodynamic Design Point \* \* \* \* \* 11 JUL 95

1 OF		
95/192		
9:16:10 Stage Temperature	VANE TO VANE	52.86
STAGE PRESSURE	RATIO VANE TO VANE	1.378
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FLOW COFFETCTENT		1.064
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9:16:10 95/192	
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11 JUL 95	S.L. NO. NO. NO. NO. NO. NO. NO. NO. NO. NO

DATA REDUCTION PAGE 4 COPY 1 OF 1

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## APPENDIX D

## LOW NOISE FAN CONFIGURATION NO. 4: AERODYNAMIC DESIGN POINT BLADE AND VANE ELEMENT PERFORMANCE AND GEOMETRY OUTPUT

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NTERSTACE AGE 10F 14.70 518.7 518.7 518.7 518.7 510.0 500.0 500.0 500.0 500.0 100.0 100.0	PERCENT 5 14.70 5 PAN 10.11 20.12 500.22 500.22 500.11 500.11 500.11 500.11 500.12 500.01 500.01
95/031 05/031 C 265/031 C 265SURE 265SURE 2610017 59119 591	RESSURE EMPERATURE VELOCITY VELOCITY 5881.6 5882.0 5882.0 5882.0 595.3 595.3 596.7 596.7 596.7
16:19:15 T0TAL PI ABSOLUTE NACH NO. 546 546 546 546 546 546 546 546 546 546	1014L 1 4850LUTE 8850LUTE 5544 5544 5549 5549 5550 5550 5550 5550
\$1 V12 MASS AVE 591:99 59 59 59 59 59 59 59 59 59 59 59 59 5	MASS AVE MASS AVE MASS AVE S811.61 5882.88 5892.00 5882.88 5893.00 5895.33 5965.33 5965.33 5966.73
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31 JAN Synthesis Annucus Corrected NCDTUS 1.8882 2.8935 3.9085 5.93215 5.93210055555555555555555555555555555555555	ANNULUS MASS FLC CORRECTION RADIUS INCHES INCHES 1.8947 1.8947 2.9911 2.9911 2.9911 2.9911 2.9911 2.9925 2.9500 2.9500 2.9500 2.9500 2.9500 2.9500 2.9500 2.9500 2.95000 2.95000 2.95000 2.95000 2.95000 2.95000 2.950000 2.95000000000000000000000000000000000000
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31 JAN STATION ANNULUS	MASS FL CORRECTI	RADIUS INCHES .884		5.971 6.977 8.983	9.985 10.987 ANNULUS	MASS FLC CORRECTE	RADIUS INCHES . 896 1.963	2.998 5.023 6.023	8.011 9.992 110.8	706.01

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31 JAN STATION ANNULUS	MASS FLO	RADIUS RADIUS 1.1229 2.189 3.182 5.1150 5.1150 5.1150 6.0125 6.0125 8.0666 8.0666 10.079	ANNULUS CARSE FLO CARSE FLO CARSE FLO TADTUS TADTUS T2.155 32.809 32.809 5.3208 5.32008 5.3208 5.32000 5.320070000000000000000000000000

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95/031 0	ESSURE MPERATURE	VELOCITY VELOCITY 675.9 676.6 684.1 684.1 684.1 684.1 703.2 703.2 708.5 708.5 708.5 708.5 708.3
6:19:15	TOTAL PRI	MARSOLUTE MACH NO. 629 633 633 642 656 656 656 662 662
slv12 1	MASS AVE. MASS AVE.	ABSOLUTE VELOCITY 675.9 679.0 684.1 689.0 689.0 689.0 703.2 708.5 708.5 708.5 708.3
* *	TED) L SQ. IN	ITURES 14711ES 14711ES 1480.6 4800.5 4779.5 1477.9 4775.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1476.5 1477.5
4 *	CORRECT	TEMPE 101701 100000000
Leaned Va sign Poin	T. 43.13 2.38 SQ. F	STATIC PRESSURE 111.25 111.25 111.18 111.14 111.04 111.04 10.95 10.95 10.95 10.95
Swept and dynamic De	RATE/SQ. F	Pressure 14.70 14.70 14.70 14.70 14.70 14.70 14.70 14.70 14.70 14.70 14.70
configure Aero	FLOW	VELOCITY VELOCITY 2597.554 250.07 250.15 202.15 130.23 103.23 2.40
Used to	re 102.78	VELOCITY VELOCITY 000000000000000000000000000000000000
95 NO. 7 EXIT 7	W RATE D FLOW RAT	VELACITY ELOCITY 6648.9 6648.9 6648.9 6646.5 6646.5 7009.4 7009.4 7009.4 7009.3 7009.3 7009.4 7009.3 7009.4
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95/031	RESSURE EMPERATURE	VELOCIT VELOCIT 6186 601105 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 6001005 60000000000	C. 7.0 Ressure Emperature	VELOCITY VELOCITY 609-60 600-14 603-72 603-7
16:19:15	TOTAL P	AB SOLUTE MACH NUTE 669 6656 6656 6556 6556 669 6681 6681 669 6656 669 6656	TOTAL PI	AB SO LUTE MACH NUTE 652 653 653 653 653 655 655 655 712 712 712 695 695
slv12	MASS AVE MASS AVE	ABSOLUTE 785.6 733.6 719.0 717.6 717.6 717.6 773.3 773.3 775.4 7776.4	MASS AVE	VERSOLUTE VERSOLUTE 7199.1 719.3 714.5 738.3 738.3 803.1 803.1 803.1 786.6
* *	FED) 9 SQ. IN	NTURES 51ATICS 5002-1-1 5022-1-1 55111-3 5525-2 552	TED)	TURES 504.1 504.1 504.1 504.1 504.1 504.1 504.1 504.1 500.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.3 520.4 520.3 520.4 520.4 520.4 520.5
ite *	T = 308.5	Т Т Т Т Т Т Т Т Т Т Т Т Т Т	CORRECT	H1222222222222222222222222222222222222
Leaned Va esign Poin	FT. 36.50 2.15 SQ. F	STATIC PRESSTI 1132.95URE 1144.865 1155.184.865 155.622 155.62	-1. 37.31 2.10 SQ.F	PRESSUR PRESSUR 133.5741 133.572 133.572 133.572 144.130 144.130 154.144 155.3111 155.3111 155.31111 155.31111 155.311111 155.31111111111
Swept and dynamic Do	RATE/SQ.	PT0TAL PTCTAL 17.753 17.751 17	RATE/SQ. F	PRESSUR PRESSUR 17.63 17.63 17.63 17.93 18.29 18.29 19.53 20.23 21.21 21.47 21.21 21.47
onfigure Aero	FLOW	VERDIAL VELOCIAT 129.19 112.77 112.77 112.75 126.25 56.16 55.16 33.031 14.65 14.65	FLOW	VERDIAL VELOCITY 85.77 75.77 7
Used to	TE 78.31	WHIRL VELOCI 4016.3 4016.3 4016.3 4019.0 4126.1 393.2 393.2	102.78 16 78.31	WHIRL VELOCITY 3999.09 3378.99 3378.99 3378.99 341.11 341.11 351.38 351.38 351.38 351.38 351.38 351.38 351.38 351.38 351.58 351.
95 NO. 9 EXIT 9	W RATE D FLOW RA	VELACIAL 602117 592.18 592.18 592.88 592.88 592.38 592.32 592.52 50 502.52 50 5	O RATE FLOW RAT	VE AXTAL AXTAL 597.15 597.15 594.01 598.05 6510.95 6630.95 6880.8 700.800
31 JAN STATION ANNULUS	MASS FLO CORRECTE	RADTUS INCHES 1NCHES 4.920 5.424 5.617 7.301 8.733 8.733 10.262 10.962	ANNULUS 1 MASS FLOI CORRECTEI	RADI INCHES 5 : 174 5

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NTERSTAGE AGE 7 OPY 1 OF	20.25	PERCENT	SPAN .5	6.⊀ -	20.9	31.0	59.0	64.4	20.00	4.66		20.25	571.5	PERCENT	SPAN	• •	12.5	21.7	21.12	53.7	64.9	87.7	4.66	
95/031 0	RESSURE EMPERATURE	MER.	VELOCITY 584.8	2.685	611.3	631.7	677.0	695.7	710.7	716.4		RESSURE	EMPERATURE	MFR.	VELOCITY	568.9	595.8	617.2	643.9	669.3	721.9	740.1	750.5	
.6: 19: 15	TOTAL P TOTAL T	ABSOLUTE	MACH NO.	. 69.5	635	.669	412	. 728	737.	.725		TOTAL	TOTAL 1	ABSOUNTE	MACH NO.	.620	630	.649	.676	00/	749	.762	.755	
slv12 1	MASS AVE. MASS AVE.	ARSOUTTE	VELOCITY	701.5	702.6	743.0	770.8	814.5	826.5	830.2		MASC AVE	MASS AVE		VELOCITY	686.4	690.5	718.9	750.7	783.5	835.6	851.1	847.2	
* * *	TED) 4 SQ.IN	ATHES	STATIC	508.2	509.7	513.4	515.6	520.3	522.8	525.4 528.3	   		O SQ. IN		STATIC	509.2	2000.5	1.112	512.4	514.0	4.712	519.4	0.125 2.72	
*	(CORREC 296.		TOTAL	548.4 549.2	550.8	559.4	565.1	570.8	579.7	582.8			CORREC		TEMPEY	548.4	549.2	550.8 554.0	559.4	565.1	570.8 77.0	2.025	582.8	
Leaned Van esign Point	FT. 38.04 2.06 SO. FT		PRESSURE	13.42	13.65	13.80	14.19	14.40	14.78	14.94	50.CT		FT. 38.61 2.03 SQ. FT		STATIC	13.60	13.63	13.69	13.90	14.03	14.18	14.44	14.56	14.00
Swept and dynamic D	RATE/SQ.		TOTAL PRESSURE	17.63	17.90	18.29	19.55	20.23	21.21	21.47	04.12		RATE/SQ.		TOTAL	PKESSUKE	17.73	17.90	18 88	19.55	20.23	21.21	21.47	21.40
onfigure Aero	FLOW	ANNU	RADIAL /FLOCITY	80.10	83.78	78.38	57.10	44.73	32.54	10.51			FLOW		RADIAL	VELOCITY	100.40	94.50	83.00	54.22	40.76	28.66	8.87	.80
Used to co	102.78	TE 78.31		389.5	380.2	374.4	391.1 407.6	420.1	423.5	413.2	393.I		102.78	AIE /8.31	WHIRL	VELOCITY	374.3	364.1	368.6	403.00	416.5	420.9	412.4	393.1
95 NO. 11	W RATE	ED FLOW RA	AXIAL VELOCITY	VELOCA 1	583.5	606.3	627.9 621.8	675.5	694.9	720.0	716.4	12	DW RATE	ED FLOW R	AXIAL	VELOCITY	559.9	588.3	611.6	640.Z	698.1	721.3	751.9	750.5
STATION	ANNULUS MASS FLC	CORRECTI	RADIUS	1045 5.045	5.307	6.264	6.872	8.188	8.869	9.557 10.254	10.965	ANNULUS	MASS FL	CORRECT	RADTUS	INCHES	5.116	5.822	6.362	6.967	, 004 8, 259	8.923	470.01	10.967

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1 1 0 2 0 31	SSURE PERATURE	VELOCITY 6542.3 6522.1 6532.1 6532.1 6532.1 7290.0 7790.0 7790.0 7790.0 7790.0 7790.0 7790.0 7790.0 7790.0 7790.0 7790.0 7779.0 77700.0 77700.0 7770.0 7770.0 7770.0 7770.0 7770.0 7770.0 7770.
6:19:15	TOTAL PRE	4850LUTE 4ACH N0. 675 675 683 .678 .721 .721 .728 .728 .739 .739 .733
slv12 1	MASS AVE. MASS AVE.	ABSOLUTE 741001174 741001174 741001174 748503 88503 88503 88798 88708 87708 877000 87700 877000 87700000000
* * *	TED) 1 SQ.IN	ATURES STATIC 502:06 500:00 500:5 50
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Leaned Van esign Point	FT. 40.11 1.95 SQ. FT	STATIC PRESSURE 112.09 113.09 113.22 113.22 113.72 114.05 114.05 114.23 14.23
Swept and dynamic D	RATE/SQ. US AREA	707AL PRESSURE 17.73 17.73 18.29 18.29 18.29 18.29 18.29 21.21 21.21 21.21 21.21 21.21 21.21
configure Aero	L FLOW	VRADIAL VELOCITY 123-085 123-085 123-085 85:73 85:73 76:85 76:85 73 73 73 73 73 73 73 73 73 73 73 73 73
Used to	IE 78.31	VMIRL VELOCITY 360.6 353.3 353.3 350.6 353.3 353.3 353.3 353.3 353.3 353.3 4113.1 4113.1 4113.1 4113.1 4113.1 333.1 4113.2 333.1 4113.2 333.1 4113.2 333.1 4113.2 333.2 4114.2 4113.2 4114.2 4114.2 4113.2 4113.2 4113.2 4114.2 4114.2 41
95 NO. 13 EXIT 13	W RATE D FLOW RAT	AXIAL 6500117 6511.2 6511.2 6511.2 6521.1 7411.7 7721.7 7721.7 7719.0 7719.0 7719.0
31 JAN STATION ANNULUS	MASS FLO CORRECTE	RADTUS INCHES 5.598 5.598 5.599 5.599 5.599 6.5099 8.916 8.916 8.916 0.286 8.962 0.2869 0.2869

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16:19:15	E. TOTAL F E. TOTAL T DIABATIC E	ABSOLUTE MACH N0. 503 512 512 512 5532 6532 6652 6652 6652 6652 6652	ETAGE ETAGE 7553 83.55 82.33 82.33 92.13 92.58 92.58 891.11 85.91 85.91	
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Used *	ATE 10	VELO	S. TE	OMEGA BAR 124 058 059 021 025 025 025 025 025 059
95 14	EXIT LOW RATE	AXIAL AXIAL 5600177 564.66 572.0 5632.0 5632.0 7564.000000000000000000000000000000000000	MACH NO EXTH NO EXTH NO 55322 55298 5652 56552 6655557 665557 665557 665557 665557 665557 6655757 665577 665577 665577 6655777 6757777 67577777777	FIFFUS FACTOR 
AL JAN	STATOR MASS FL CORRECT	RADIUS INCHES 5.5595 5.5595 5.5595 6.230 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2300 7.2000 7.20000 7.20000 7.20000000000	ABSOLUTE INLET 6683 6683 721 721 721 721 721 721 721 721 721 721	21975111973575 21975111973575 219753119735

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* * *	CTED)	RATURES STATIC 527.3	526.9 526.9 527.3	532.7 531.4	534.2	CTED) .3 SQ.IN	RATURES 579.9 529.9 528.6 528.6	5227.09 5228.02 5228.02 5238.02 5331.01 5331.0
r e	CORRE	TEMPE TOTAL 548.4	0400 0400 0400 0400	575.55 579.55 5829.75	583.9	CORRE	TEMPE 5548.45 550.28 550.28 550.28 550.28	8874084 8874084 8874084
i Leaned Va Jesign Poin	FT. 40.78 1.95 SQ. F	STATIC STATIC PRESSURE 14.86	15.08 15.18	11155 1155 1155 1155 1155 1155 1155 11	15.36	FT. 40.64 1.95 SQ. F1	STATIC PRESSURE 15.13 15.16 15.20	1125.24 1
swept and odynamic [	LUS AREA	TOTAL PRESSURE 17.06 17.23	17.51 17.99 18.66 19.37	20.05 20.58 21.195	20.98	RATE/SQ. LUS AREA	T0TAL PRESSURE 17.06 17.23 17.51 17.51 18.66	20.05 20.05 20.05 21.195 20.98
configure Aer	B FLOW	VELOCITY VELOCITY 12.63 15.20	17.97 19.57 19.05 16.24	11.85 6.62 - 4.07	n	FLOW	VELOCITY VELOCITY 3.27 5.48 7.06 7.52	6.79 5.18 3.04 2.13 - 1.15
Used to	E 79.3	WHIRL	وووو	وووو	2	102.78	WHIRL	000000
95 NO. 15 EXIT 15	DW RATE	VELOCITY 503.9 516.5	572.0 619.9 666.9	772.9 775.9	9	W RATE D FLOW RATE	AXIAL VELOCITY 471.4 489.4 516.3 560.0 515.0	667.9 7414.6 7728.3 785.9 786.9
31 JAN STATION ANNULUS	MASS FLC CORRECTE	RADIUS INCHES 5.669 5.914	567.9 567.9 567.9 567.9	9.695 9.695 10.317 10.963	ANNULUS 1	MASS FLO CORRECTE	RADIUS 55.9883 55.9883 55.9883 56.3335 56.3335 58275 5875 587555 587555 5875555 5875555 5875555 5775555 5775555555	10.961 10.317 10.317
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1 95/031	RESSURE EMPERATURE	VE MER. 4ELOCITY 481.0 5510.2 5510.2 5510.2 66136.5 66136.5 66136.5 717505.5 7506.8 7874.9 7874.9 787.3 777.3 7777.3 777.3 777.3 777.3 777.3 777.3 7777.3 7777.3 7777.3 7777.3	RESSURE EMPERATURE MER MER 458.8 458.8 653.3 555.3 555.3 555.3 718.0 718.0 772.8 781.2 781.2
16:19:15	TOTAL P	ABSOLUTE MACH NO. .426 .426 .426 .534 .536 .536 .536 .687 .687 .687 .689	. T01AL P . T01AL P . T01AL T . T01AL T . T01AL T . 5853 . 5845 . 5845 . 5889 . 6689 . 6689 . 690 . 690
slv12	MASS AVE	ABSOLUTE VELOCITY 481.0 510.2 510.2 516.5 516.5 668.6 668.6 774.9 787.7 780.3 780.3	MASS AVE MASS AVE MASS AVE AEBCOLITE 4613.66 613.66 613.66 613.67 752.88 781.2 781.2
* * * *	TED) 6 SQ.IN	ATURES S300.7 5229.10 5228.44 5228.64 5228.66 5228.66 5228.66 531.22 531.22	CTED) 6 5Q. IN 8771C 530.19 5228.55 5227.88 5227.88 5227.88 5227.88 5321.0
<b>*</b>	(CORREC	T1420202020202020202020202020202020202020	Т СО СО СО СО СО СО СО СО СО СО СО СО СО
Leaned Van esign Point	FT. 40.60 1.96 SQ. FT	PRESSUR PRESSUR 15:22 15:22 15:22 15:22 15:22 15:22 23 23 23 23 23 23 23 23 23 23 23 23 2	FT 1.96 80.60 815 815 815 815 815 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 825 115 115 115 115 115 115 115 1
Swept and odynamic Do	RATE/SQ.	PRESSAL PRESSAL 177.23 177.23 117.53 117.53 119.666 20.538 20.558	LUS AREA LUS AREA TOTAL PRESSURE 17.51 17.53 17.
nfigure Aero	FLOW	RADIAL READIAL 1.20 1.50 1.50 1.50 1.50 1.96	Provide the second seco
Used to co	102.78		TE 102.78 WHIRL 79.39 WHIRL VELOCITY .00 .00 .00 .00 .00 .00
1 95 17_	S EXIT 17	ED FLOW AS AXIAL AXIAL 4601177 4601177 4811.05 5510.22 7150.85 787.79 780.33 780.33	18 FED RATE AXIAL VELOUK R 458:8 5558:3 555855755757557575757575757575757575757
31 JAN STATION	ANNULUS MASS FL	CORRECT RADIUS INCHES 5:938 6:333 6:333 6:333 6:333 6:333 7:938 8:535 10:315 10:358	ANNULUS CARSE FI CARSE FI CARS

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UMMARY 1 1 OF 1		
PERF. S PAGE COPY	FLOW COEFF. 1.145	HORSE POWER 1844.4
150 95/031	SPECIFIC FLOW IN 0UT 43.13 35.58 40.11 41.08	INLET AXIAL VELOCITY MEAN 686.3 718.7
v12 16:19	MACH NO. 1.144 -	TOTAL TURNING HUB TIP 27.4 10.7 29.7 27.0
[5 * * * *	LOAD COEFFICIENT (MEAN WHEEL SPEED) HUB MEAN TIP .497 .776 1.091	FLOW ANGLE FLOW ANGLE HUB TIP -3.4 43.9 .0
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APPENDIX E

## STRUCTURAL ANALYSIS RESULTS ROTATING COMPONENTS

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APPENDIX F

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RESULTS OF DYNAMIC ANALYSIS BLISK

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Figure A32: Dynamic Stress Plot of Blade Mode 1 - Suction Side


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## APPENDIX G

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## STRESS ANALYSIS RESULTS STATIC STRUCTURE

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Ċ LOAD SET LEGEND \*\* KS I 67 00 57 00 47 00 37 00 27 00 17 00 67 89 67 89 updated 2/8/95 A C C C C C C C F F G G F A MAX \* MIN \* MIN \* \* \* PLOT TIME AND DATE = 19:13:26 95/039 TITLE NASA rig w/ baseline vane: vane + AOA + weight; CONTOUR PLOT OF VON MISES UNIAXIAL EQUIVALENT STRESS .1000 SCALE =



3 LOAD SET \* \* \* 57.00 47.00 37.00 27.00 17.00 67.00 7.00 00 67.89 H I DDEN KS I updated 2/8/95 LEGEND \* DENOTES \* \* \* へきじつきょう NIN \* \* MAX 95/039 TITLE NASA rig w/baseline vane: vane + AOA + weight; VON MISES UNIAXIAL EQUIVALENT STRESS PLOT TIME AND DATE = 19:15:49 CONTOUR PLOT OF .1300 SCALE =







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\_ LOAD SET \* \* 35.00 27.00 19.00 00 43.00 3.00 00. H IDDEN 59.00 11.00 59.38 KS I 3/07/95 LEGEND \* 4 PLOT TIME AND DATE = 16:10:46 95/066 CONTOUR PLOT OF VON MISES UNIAXIAL EQUIVALENT STRESS 2/ TITLE NASA rig w/ aft vane: vanc loads .1300 SCALE =






















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ĉ LOAD SET \* \* 2/11/95 u p d a t e d TITLE NASA rig w/ swept & leaned vane: vane + AOA + weight PLOT TIME AND DATE = 09:15:38 95/042 CONTOUR PLOT OF VON MISES UNIAXIAL EQUIVALENT STRESS Į 0660. SCALE =

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## APPENDIX H

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## STATIC STRUCTURE DEFLECTION ANALYSIS RESULTS

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c LOAD SET updated 2/11/95 TITLE NASArig w/swept & leaned vane: vane + AOA + weight PLOT OF DEFLECTED SHAPE SCALE = 1300 PIOT TIME \*\*\*\* 27



APPENDIX I

## RESULTS OF DYNAMIC ANALYSIS OF FULL NACELLE SYSTEM

















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APPENDIX J

## **RESULTS OF DYNAMIC ANALYSIS VANE MODES**

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гo pressure side 95/207 00. 21 11 PLOT TIME AND DATE = 18:07:00 SECTOR PATTERN TITLE NASA scaled fan rig - swept & leaned vane PLOT OF NODE LINE NORMAL TO VIEWING PLANE = IHd 6041.609 10 .6500 MODE NUMBER = FREQUENCY = SCALE = L -



## Appendix C: Dynamic Stress Contour Plots

List of Figures

C1 - C2: Baseline & Aft Vanes - Mode 1 (1B) C3 - C4: Baseline & Aft Vanes - Mode 2 (1T) C5 - C6: Baseline & Aft Vanes - Mode 3 (2B) C7 - C8: Baseline & Aft Vanes - Mode 4 (2T) C9 - C10: Baseline & Aft Vanes - Mode 5 (3B) C11 - C12: Baseline & Aft Vanes - Mode 6 (3T) C13 - C14: Baseline & Aft Vanes - Mode 7 C15 - C16: Baseline & Aft Vanes - Mode 8 C17 - C18: Baseline & Aft Vanes - Mode 9 C19 - C20: Baseline & Aft Vanes - Mode 10 C21 - C22: Swept Vane - Mode 1 (1B) C23 - C24: Swept Vane - Mode 2 (1T) C25 - C26: Swept Vane - Mode 3 (2B) C27 - C28: Swept Vane - Mode 4 (2T) C29 - C30: Swept Vane - Mode 5 (3B) C31 - C32: Swept Vane - Mode 6 (3T) C33 - C34: Swept Vane - Mode 7 C35 - C36: Swept Vane - Mode 8 C37 - C38: Swept Vane - Mode 9 C39 - C40: Swept Vane - Mode 10 C41 - C42: Swept & Leaned Vane- Mode 1 (1B) C43 - C44: Swept & Leaned Vane- Mode 2 (1T) C45 - C46: Swept & Leaned Vane- Mode 3 (2B) C47 - C48: Swept & Leaned Vane- Mode 4 (2T) C49 - C50: Swept & Leaned Vane- Mode 5 (3B) C51 - C52: Swept & Leaned Vane- Mode 6 (4B) C53 - C54: Swept & Leaned Vane- Mode 7 C55 - C56: Swept & Leaned Vane- Mode 8 C57 - C58: Swept & Leaned Vane- Mode 9 C59 - C60: Swept & Leaned Vane- Mode 10

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suction side 95/207 00 21 H 18:10:23 PATTERN STRESS SECTOR PLOT TIME AND DATE = = IHd 2559.022 .6100 11

\* 90.00 70.00 50.00 50.00 31.00 20.00 100.00 \* DENOTES HIDDEN 100.00 10.00 PS I LEGEND \* MAX \* MIN **A B C C B F** σ Ξ \* CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC TITLE NASA scaled fan rig - swept vane MODE NUMBER = FREQUENCY SCALE =

pressure side 95/207 00 21 80.00 70.00 60.00 40.00 30.00 10.00 H I DDE N 100.00 90.00 . 0 5 00.00 20.00 PS I LEGEND PATTERN = 18:09:13 \* MIN \* DENOTES CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS \* MAX **A U O U U F O H** \* \* \* SECTOR PLOT TIME AND DATE = PH1 = TITLE NASA scaled fan rig - swept vane 3167.500 ŝ .6100 II MODE NUMBER п FREQUENCY SCALE =





suction side \* \* \* 95/207 80.00 70.00 50.00 40.00 20.00 10.00 100.00 00 \* DENOTES HIDDEN 00.00 30.00 . 01 2 1 00 PS I .06 LEGEND SECTOR PATTERN = 18:10:39 CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS \* MAX \* MIN \* **< B C D E C H** \* \* \* PLOT TIME AND DATE = = IHd TITLE NASA scaled fan rig - swept vane 6 3889.117 .6100 11 MODE NUMBER FREQUENCY = SCALE =

· pressure side 95/207 80.00 70.00 60.00 50.00 30.00 30.00 10.00 .04 \* DENOTES HIDDEN 21 000. 100.00 90.00 PS I LEGEND 11 18:09:29 SECTOR PATTERN CONTQUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS \* MAX \* MIN **AUDUFOH** PLOT TIME AND DATE = = 1 H dTITLE NASA scaled fan rig - swept vane 4001.385 5 . 6100 MODE NUMBER = 11 FREQUENCY SCALE =

suction side \* \* \* 95/207 90.00 80.00 60.00 50.00 40.00 20.00 20.00 10.00 MAX 100.00 \*MIN .04 \*DENOTES HIDDEN 00. 21 PS I LEGEND H 18:10:49 SECTOR PATTERN CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS **A B D D B F D H** \* \* \* PLOT TIME AND DATE = PHI = TITLE NASA scaled fan rig - swept vane 4001.385 ~ . 6100 MODE NUMBER = FREQUENCY = SCALE =









pressure side 95/207 \* 0 2 H I DDE N 00 70.00 60.00 50.00 40.00 30.00 21 80.00 20.00 10.00 100.00 90.00 # PS LEGEND R 18:09:48 SECTOR PATTERN \* DENOTES CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS NIN \* **ABCUBF**OH \* MAX \* \* \* PLOT TIME AND DATE = = IHd TITLE NASA scaled fan rig - swept vane 0 6353.275 .6100 u MODE NUMBER FREQUENCY = SCALE =



LO pressure side \* \* 100.00 30.00 \* DENOTES HIDDEN 70.00 60.00 . 02 PSI 90.00 80.00 20.00 50.00 40.00 95/207 00. 2.1 LEGEND SECTOR PATTERN = PLOT TIME AND DATE = 18:08:43 CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS TITLE NASA scaled fan rig - swept & leaned vane NIM \* MAX < m O D B L O H -\* \* \* = 1 H d ĥ 636.257 . 6500 11 MODE NUMBER FREQUENCY = SCALE =


pressure side \* 50.00 40.00 30.00 20.00 02 H IDDEN 10.00 00.00 80.00 70.00 60.00 00.00 90.00 95/207 PSI 00 21 LEGEND \* DENOTES ti 18:08:50 SECTOR PATTERN NIN \* TITLE NASA scaled fan rig : swept & leaned vane CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS **HOTEDCEA** \* \* \* PLOT TIME AND DATE = PHI =1428.280 2 . 6500 8 MODE NUMBER 11 FREQUENCY SCALE =

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LO pressure side 100.00 90.00 80.00 70.00 60.00 50.00 30.00 20.00 100.00 \* DENOTES HIDDEN 0. 95/207 PS I 00. 21 LEGEND 11 18:09:02 SECTOR PATTERN CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS NIM \* TITLE NASA scaled fan rig - swept & leaned vane MAX **A B C D B F C H** \* \* \* PLOT TIME AND DATE = PHI = 2545.286 .6500 11 MODE NUMBER H FREQUENCY SCALE =

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LO suction side \* \* \* 100.00 90.00 80.00 60.00 50.00 50.00 50.00 20.00 10.00 \* MAX 100.00 • MIN 02 \* DENOTES HIDDEN PS I LEGEND 95/207 00. 21 II 18:10:27 **A B C D B F** \* \* \* UH SECTOR PATTERN TITLE NASA scaled fan rig - swept & leaned vane MAXIMUM PRINCIPAL DYNAMIC STRESS PLOT TIME AND DATE = = IHd 7 ŝ 3007.614 CONTOUR PLOT OF . 6500 11 MODE NUMBER Ħ FREQUENCY SCALE =

го pressure side 4 \* \* 90.00 80.00 70.00 60.00 50.00 40.00 30.00 20.00 10.00 100.00 \* DENOTES HIDDEN 100.00 PS I 95/207 LEGEND 00. 21 11 18:09:15 SECTOR PATTERN < mOOm LOH \* MAX TITLE NASA scaled fan rig - swept & leaned vane CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS \* \* \* PLOT TIME AND DATE = = IH4 9 3603.865  $\hat{\mathbf{b}}$ .6500 u MODE NUMBER ŧI FREQUENCY SCALE = Z Σ

го suction side \* \* \* 70.00 60.00 50.00 40.00 . 03 H I DDEN 20.00 10.00 00.00 100.00 90.00 80.00 PS I 95/207 LEGEND 00. 21 \* DENOTES H PLOT TIME AND DATE = 18:10:34 NAX MIN SECTOR PATTERN **ABODBFOH** \* \* \* TITLE NASA scaled fan rig - swept & leaned vane CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS = IHd Σ 3603.865 9 .6500 MODE NUMBER = u FREQUENCY SCALE =



L0 suction side \* \* \* 50.00 40.00 30.00 20.00 • MIN . 03 • DENOTES HIDDEN 03 70.00 10.00 100.00 80.00 00.00 90.00 95/207 PS 1 LEGEND 00. 21 SECTOR PATTERN = 18:10:42 MAX **ABODBFOH** TITLE NASA scaled fan rig · swept & leaned vane CONTOUR PLOT OF MAXIMUM PRINCIPAL DYNAMIC STRESS \* \* \* PLOT TIME AND DATE = = IHd-4035.621 .6500 MODE NUMBER = FREQUENCY = SCALE =



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