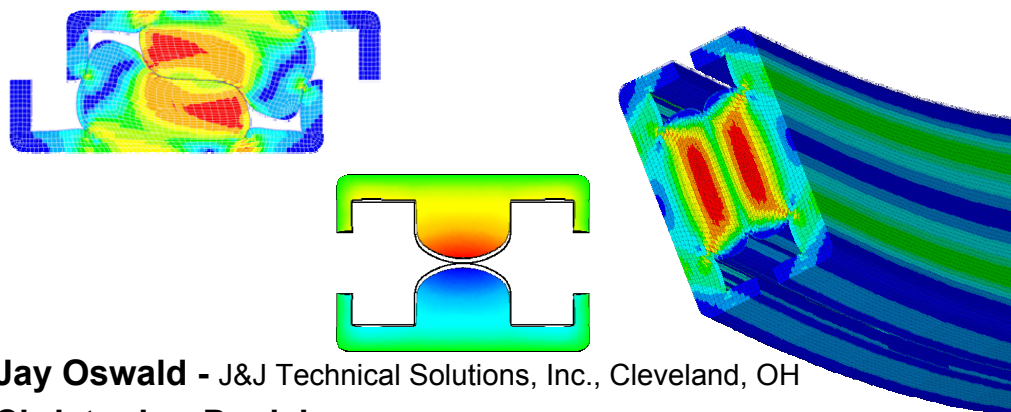


FINITE ELEMENT ANALYSIS OF ELASTOMERIC SEALS FOR LIDS

Jay J. Oswald
J&J Technical Solutions, Inc.
Cleveland, Ohio

Christopher C. Daniels
University of Akron
Akron, Ohio

Finite Element Analysis of Elastomeric Seals for LIDS



Jay Oswald - J&J Technical Solutions, Inc., Cleveland, OH

Christopher Daniels - The University of Akron, Akron, OH

Prepared for the 2006 NASA Glenn Seals/Secondary Air System Workshop
November 15, 2006

Objectives & Motivation

Objective

- Create a means of evaluating seals w/o prototypes

Motivation

- Cost
 - Prototype 54” seal ~\$100k per seal pair
 - FEA license + high end workstation ~ \$30k per year
- Development time
 - 6 months lead time for a new seal design
 - Many designs per day (solution time <1 minute)
- Understanding
 - Difficult to experimentally measure strains, contact pressure profile, stresses, displacements

Part I

Hyperelastic Material Modeling

Special Properties of Hyperelastic Materials

- Fully or nearly Incompressible
 - Bulk modulus typically 100-1000x shear modulus
 - Poisson's ratio approaches 0.5
 - Problems in displacement-based FEA formulation
 - Requires B-bar or mixed u-P formulation
- Huge elastic range of deformation
 - Strains > 80% are (mostly) recoverable
 - Analysis should account for nonlinear geometry and material properties

Hyperelasticity vs. Linear Elasticity

Linear elasticity:

$$\mathbf{W} = \mathbf{C}\boldsymbol{\varepsilon}:\boldsymbol{\varepsilon}$$

(which is like: $E = \frac{1}{2} k\Delta x^2$)

Hyperelasticity:

$$\mathbf{W} = \mathbf{f}(I_1, I_2, I_3)$$

$$\text{or } \mathbf{W} = \mathbf{f}(\lambda_I, \lambda_{II}, \lambda_{III})$$

Definition of second Piola-Kirchoff stress from strain energy density and Green-Lagrange strain

$$\sigma_{ij} = \frac{\partial W}{\partial \varepsilon_{ij}}$$

$$I_1 = \lambda_I^2 + \lambda_{II}^2 + \lambda_{III}^2$$

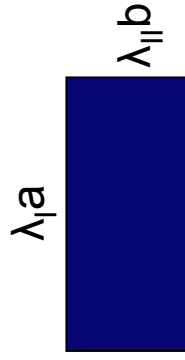
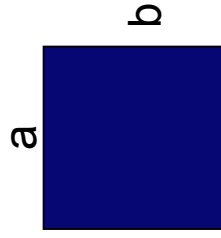
$$I_2 = \lambda_I^2 \lambda_{II}^2 + \lambda_{II}^2 \lambda_{III}^2 + \lambda_{III}^2 \lambda_I^2$$

$$I_3 = \lambda_I^2 \lambda_{II}^2 \lambda_{III}^2 = 1 + \left(\frac{\Delta V}{V} \right)^2 = J^2$$

$\lambda_I, \lambda_{II}, \lambda_{III}$: principal stretch ratios

I_1, I_2, I_3 : strain invariants

J : Jacobian (volume ratio)



Some forms of the work function

Polynomial models: (Mooney-Rivlin, Neo-Hookean)

$$W = \sum_{i+j=1}^N C_{ij} (\bar{I}_1 - 3)^j (\bar{I}_2 - 3)^i + \sum_{k=1}^N \frac{1}{d_k} (J - 1)^{2k}$$

Yeoh model: $j=0$, neglects second strain invariant

- For plane strain Yeoh is equivalent to general polynomial form because $I_1 = I_2$

Comparison of lowest order terms for a 50 durometer material

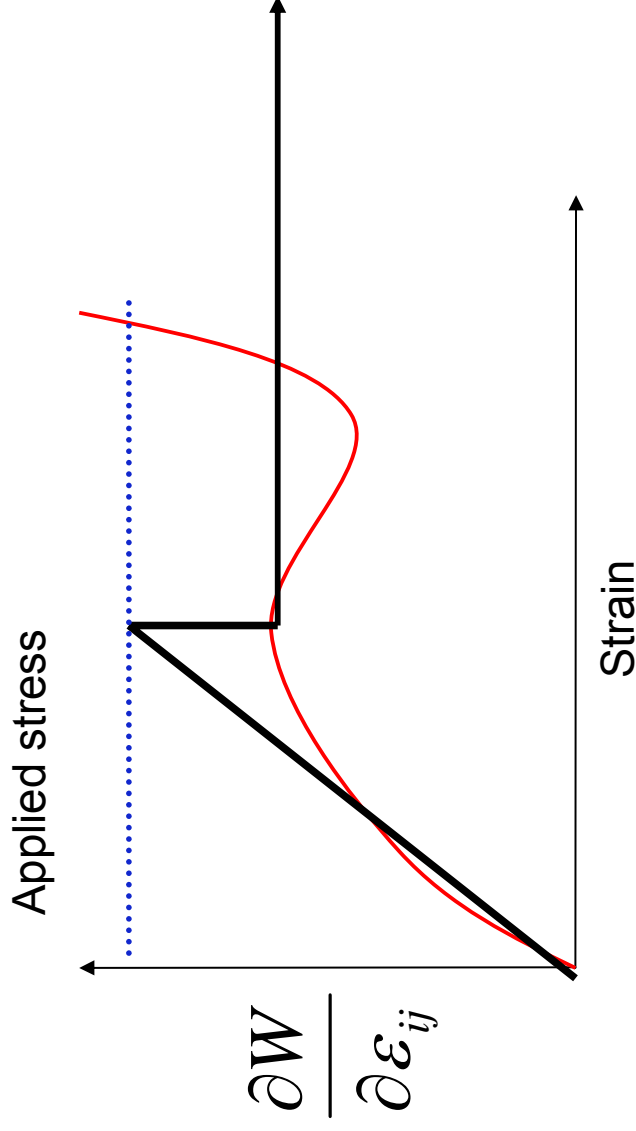
$$1/d_1 \approx 200,000 \qquad C_{1,0} \approx 40$$

Constraints on the work function

Zero strain must have zero energy ($W(0) = 0$)

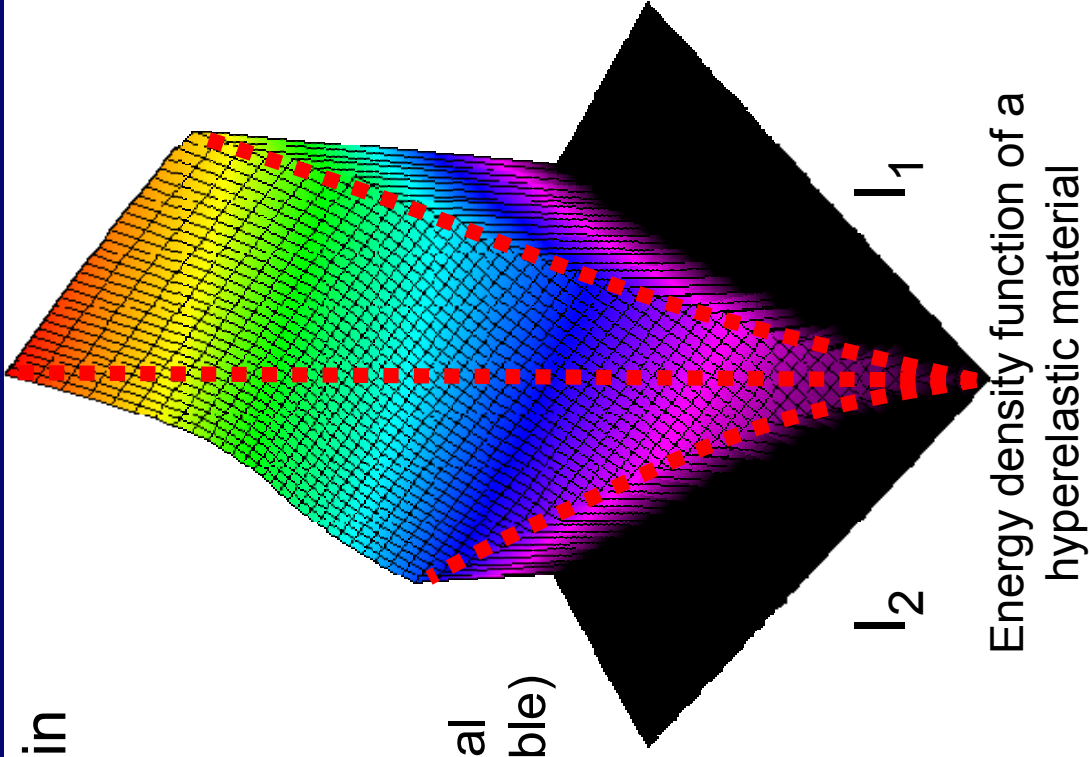
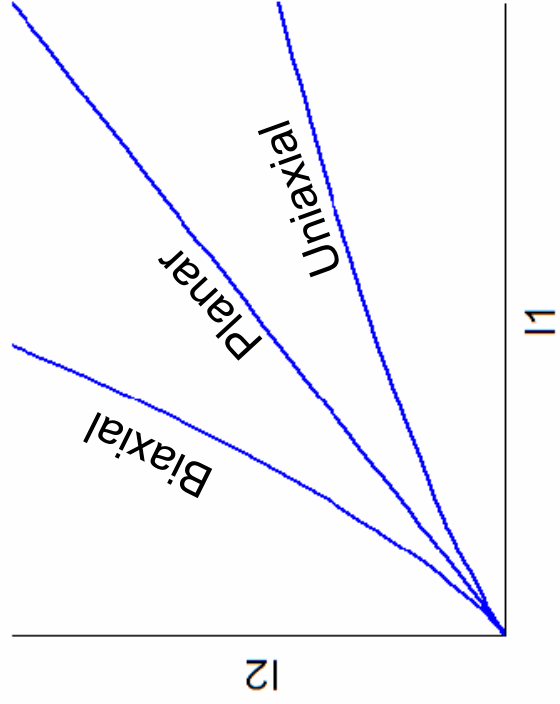
Zero strain must have zero stress ($W'(0) = 0$)

Second derivative must be positive ($W''(\epsilon) > 0$ for all ϵ)

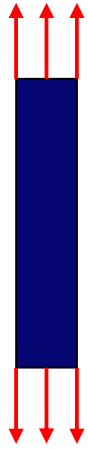
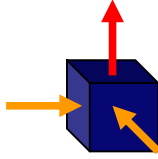
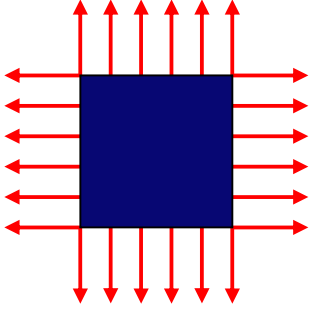
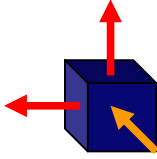
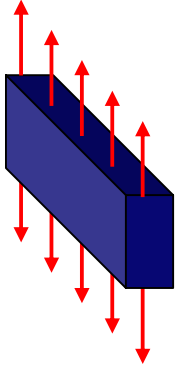
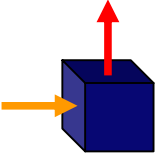
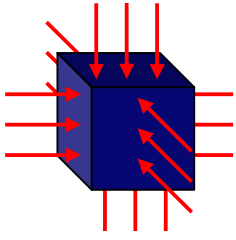
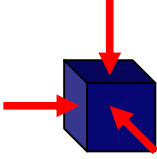


Determining W

- Fit W to experimental stress-strain states
 - Three basic strain modes
 - Uniaxial tension
 - Biaxial tension
 - Planar tension
 - All deformation falls between uniaxial and biaxial – ($I_3 = 1 \rightarrow$ incompressible)



Basic strain states of a nearly incompressible material

	Load	Strain	Stretch Ratios
Uniaxial			$\lambda_I = \frac{1}{\lambda_{II}^2} = \frac{1}{\lambda_{III}^2}$
Biaxial			$\lambda_I = \lambda_{II} = \frac{1}{\sqrt{\lambda_{III}}}$
Planar			$\lambda_I = \frac{1}{\lambda_{II}}, \lambda_{III} = 1$
Volumetric			$\lambda_I = \lambda_{II} = \lambda_{III} < 1$

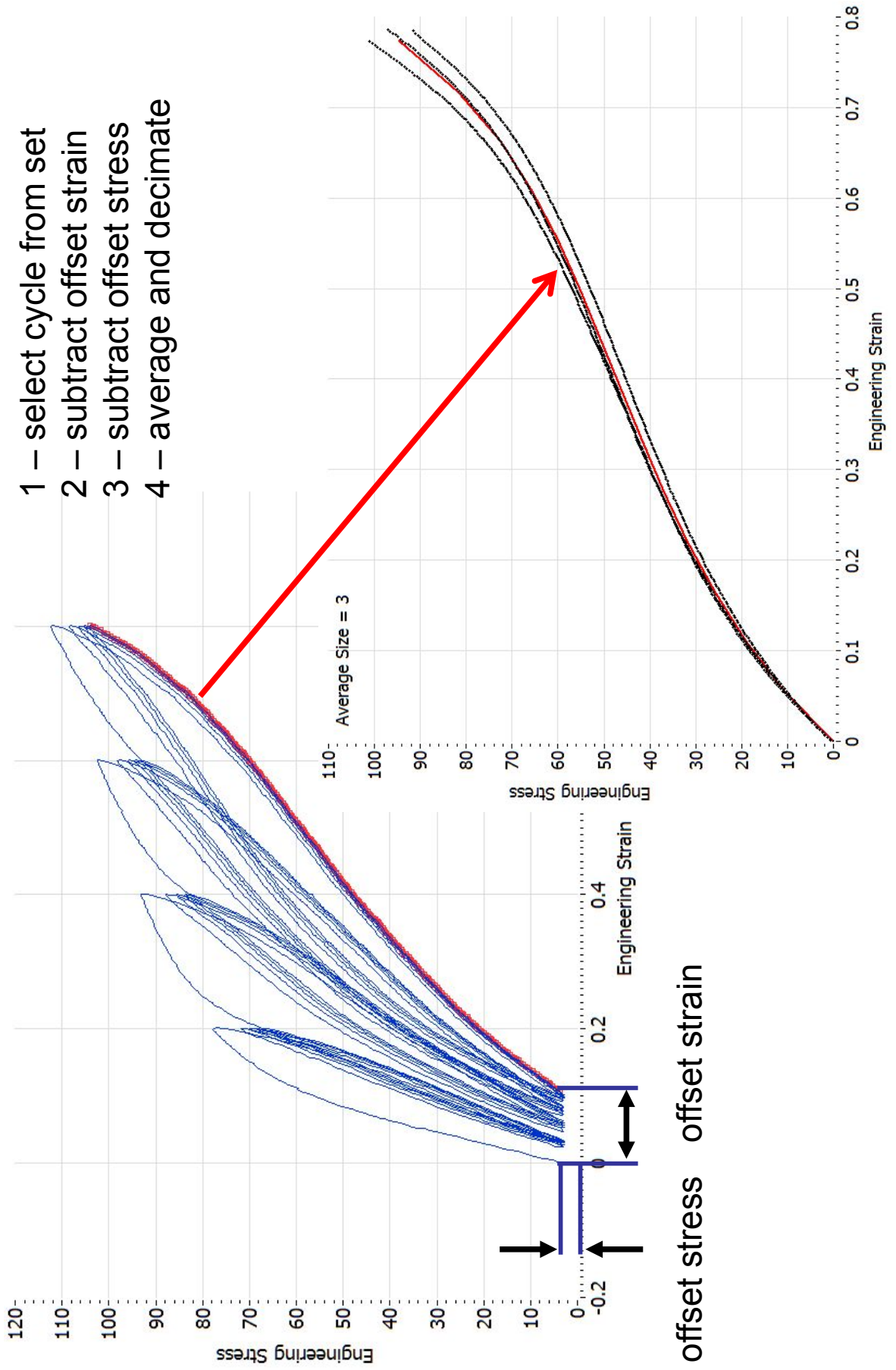
Material Tests Performed

- **Materials:** XELA-SA-401, S0899-50, S0383-70
 - 40, 50, 70 durometer hardness
- **Test parameters**
 - Various temperatures
 - -50, 23, 50, & 125 °C
 - 3 specimens per test
 - Uniaxial, planar, biaxial tension & volumetric
 - 20, 40, 60, 80 % strain increments
- **Other properties:**
 - Coefficient of friction (elastomer on elastomer), thermal conductivity, heat capacity, density, emissivity, absorptivity

This data will be published soon in a NASA technical publication

Data Processing

- 1 – select cycle from set
- 2 – subtract offset strain
- 3 – subtract offset stress
- 4 – average and decimate

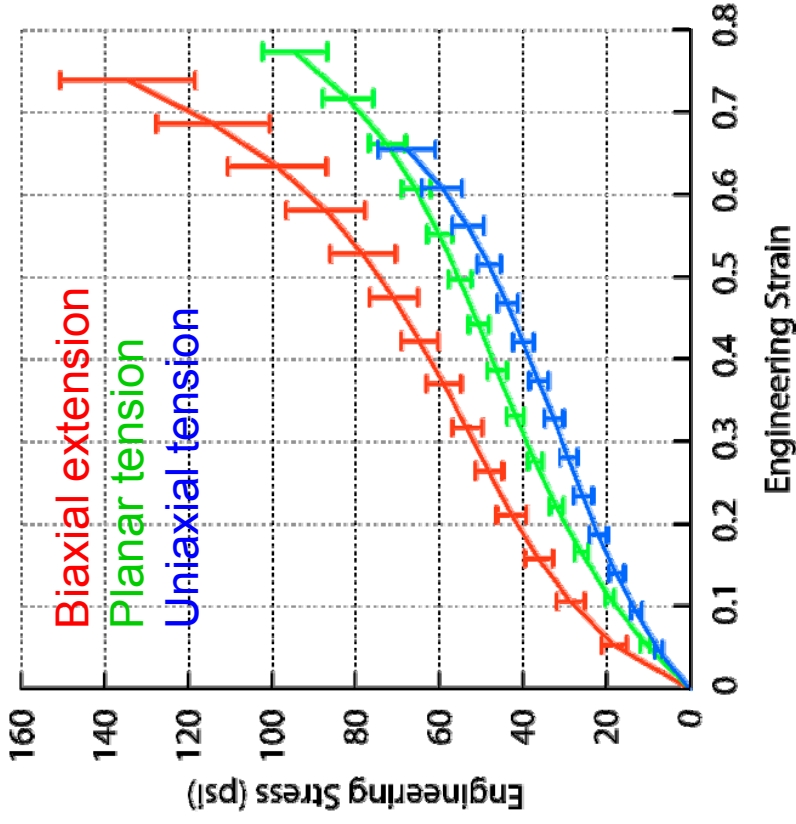


Processed Material Data

Uncertainty based off student's t distribution from multiple specimens

Results can be curve fitted to determine material property constants

This can be done as a function of temperature



Processed data from 40 durometer elastomer (-50 °C)

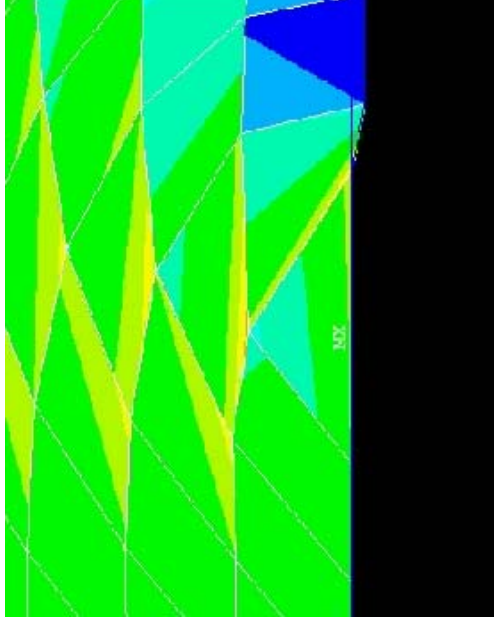
The strain energy density is the area under the curve for each deformation

Part II

Finite Element Analysis of Seals

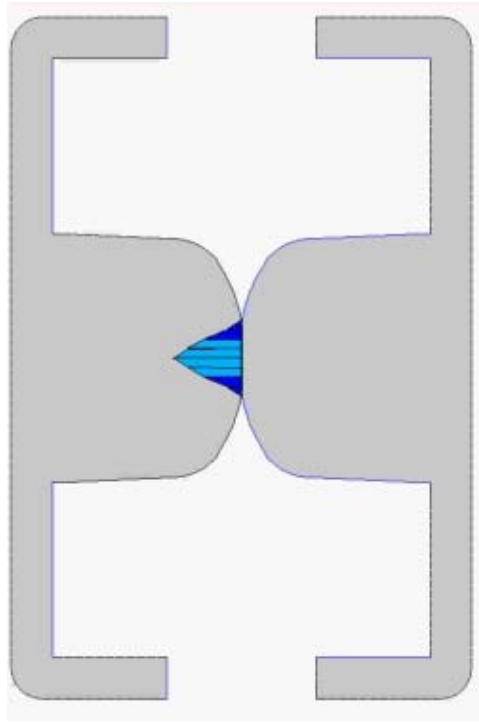
Hints for Elastomeric FEA

- 1) Stay away from triangular elements
 - Elements with 2 displacement BC will have only 1 degree of freedom due to incompressibility
- 2) Low order elements converge easiest 4-node brick works well
- 3) Sliding contact may require non-symmetric stiffness matrices for large friction coefficients
- 4) Watch corners for element distortion
- 5) u-P element formulation is most stable
- 6) Check for stability of material models



Severe element edge distortion
Analyses did not converge

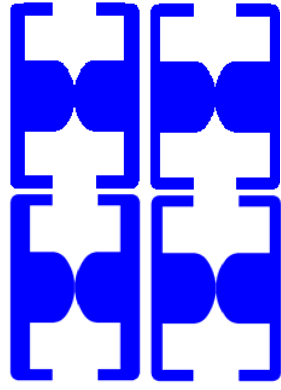
Types of FEA models of LIDS seals



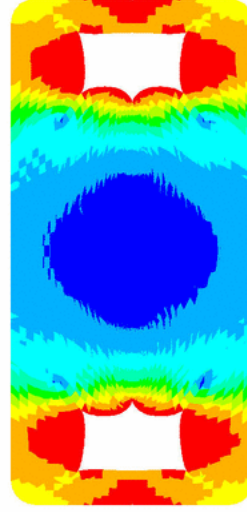
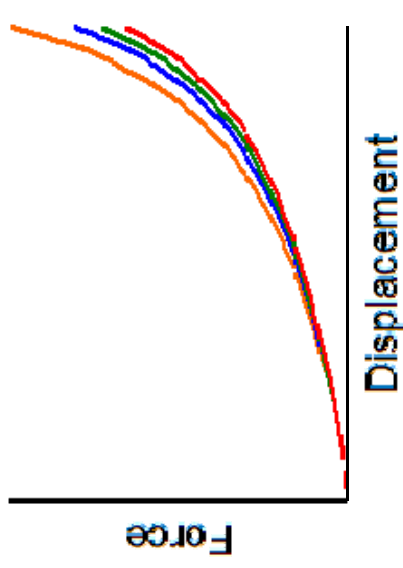
Aligned seal – contact pressure



Misaligned seal
Principal strains



Tolerance studies



Gaseal adhesion analysis with
cohesive elements at contact

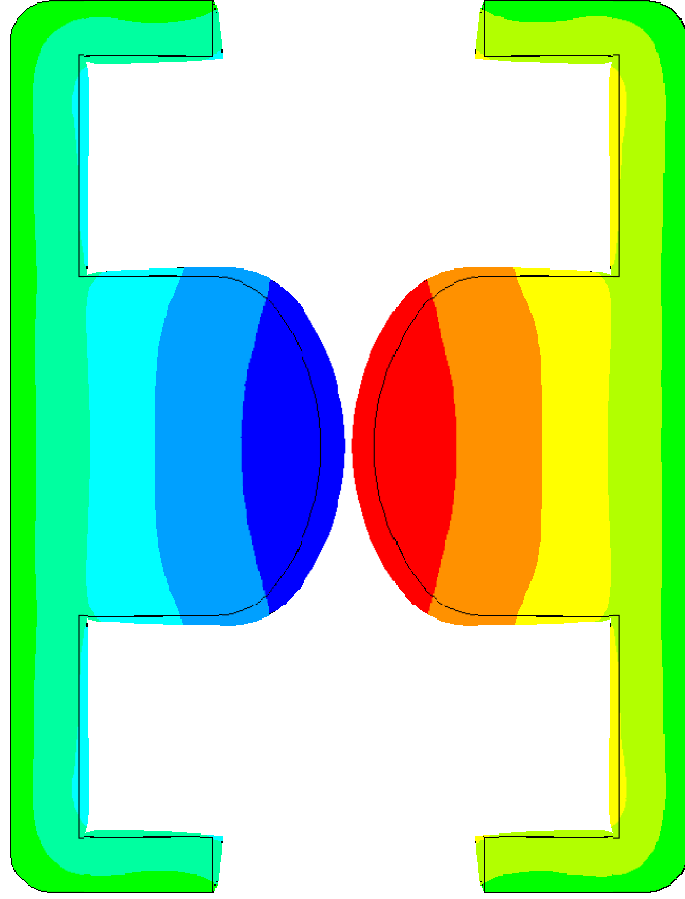
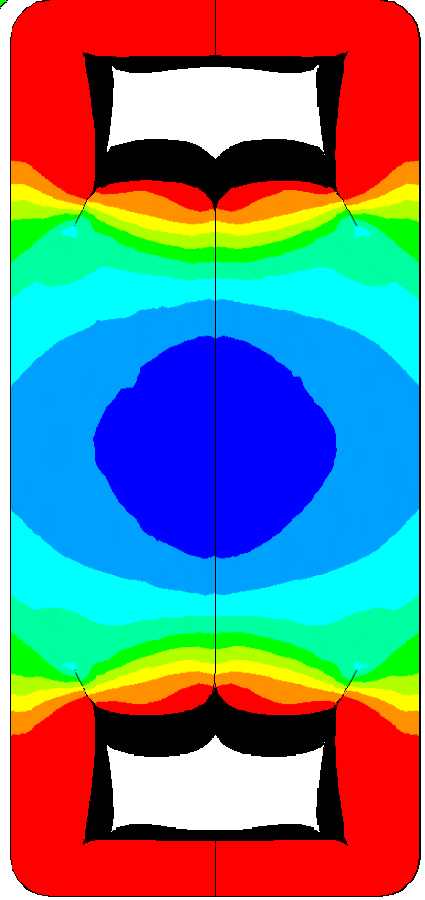
Seal Thermal Analyses

- CTE of elastomers is very high

- $350 \times 10^{-6} \text{ } ^\circ\text{C}^{-1}$

- Al: $24 \times 10^{-6} \text{ } ^\circ\text{C}^{-1}$

Comparison of compression at 25°C (front) and 125°C (back). Contours are axial stress.



Y displacement of seals with 100°C rise in temperature, black outline indicates original geometry

Summary

- Need 4 experimental strain states to
 - choose energy density function
 - fit material constants
 - determine compressibility of material
- Hyperelastic material present new challenges
- FEA analyses for LIDS
 - Force vs. displacement and pressure contours
 - Aligned & misaligned cases
 - Thermal expansion
 - Tolerance studies
 - Adhesion analysis

Further reading/information

- ANSYS gives excellent background for element technology/hyperelasticity
 - Nonlinear element technology
 - <http://www.ansys.com//assets/tech-papers/nonlinear-element-tech.pdf>
 - Hyperelasticity
 - http://www.tsne.co.kr/board/download.asp?strFileName=conflong_hyperel.pdf&dr=ansys
- Future publications of material properties, analysis, etc. will be posted on <http://www.grc.nasa.gov/WWW/structuralseal>