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Amplitude Scaling of Active Separation Control

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Motivation



- Active control of boundary layer separation leads to many benefits
- Frequency parameter: Strouhal (F⁺)
- Length scale under controversy
- Amplitude scaling?
- We use Cµ historically, steady blowing
- But does it scale the data?
- Are there other parameters we should consider?
- We revisit here known and define 2 new amplitude scaling parameters
- Check their validity using low Re data (Re<1x10⁶)











Amplitude scaling - 1



Velocity Ratio

 $VR \equiv \frac{U_p}{U_e} \approx \frac{U_p}{U_\infty}$

- U_p- peak excitation velocity
- U_e local external velocity
- U_{∞} free-stream velocity
- Actuator Mach number over freestream Mach number
- Implies linearity







- Momentum coefficient
- Ratio between (incomp., isoth.)
 - Excitation momentum (*h*=slot width)
 - Free stream momentum (*c*=chord)
- U_p peak excitation velocity
- U_{∞} free-stream velocity
- h slot width, c airfoil chord
- Follows steady-blowing concept

(Yehoshua)

 $C_{\mu} \equiv \frac{h}{c} \left(\frac{U_{p}}{U_{c}} \right)^{2} = \frac{h}{c} (VR)^{2}$







Amplitude scaling - 3

 $\approx h U_p^2$

 $\approx \theta U_{\infty}^2$



Reynolds # corrected momentum coefficient

- Ratio between
 - Excitation momentum
 - BL momentum deficit
 - Laminar momentum thickness
 - Turbulent momentum thickness

$$\frac{\theta_{Lam}}{c} \approx \text{Re}^{-0.5}$$

$$\frac{\theta_{Turb}}{c} \approx \operatorname{Re}^{-0.2} \Longrightarrow \theta_{Turb} \approx c \operatorname{Re}^{-0.2}$$

$$C_{\mu,\text{Re}} \equiv \frac{h}{\theta} \frac{U_p^2}{U_\infty^2} \approx \frac{h(VR)^2}{c \,\text{Re}^m} = \frac{h}{c} (VR)^2 \,\text{Re}^{-m}$$



Vorticity-flux / Circulation - 4



 Actuator's vorticity flux (blowing, half slot width)

$$VF \approx \int_{0}^{\frac{T}{2}/2} \int_{0}^{h/2} \omega_{z}(y,t) U_{p}(y,t) dy dt$$

- Kutta-Zhukovsky
- Circulation
- Vorticity-flux coefficient

$$\begin{split} VF &\approx \int_{0}^{h/2} \int_{0}^{T/2} U_{p}(y,t) \frac{dUp(y,t)}{dy} dt dy \\ VF &\approx \frac{U_{p}^{2}}{h} \frac{hT}{4} \approx \frac{U_{p}^{2}T}{4} \approx \frac{U_{p}^{2}}{4f} \\ C_{L} &= \frac{\rho_{\infty} U_{\infty} \Gamma}{1/2 \rho_{\infty} U_{\infty}^{2} c} \\ \Gamma &= cC_{L} U_{\infty} \\ C_{\Gamma} &\equiv \frac{Slot_Vorticity_Flux}{Baseline_Circulation} \approx \frac{U_{p}^{2}T}{4\Gamma_{Baseline}} \\ C_{\Gamma} &\approx \frac{U_{p}^{2}T}{4cC_{L} U_{\infty}} = \frac{1}{4C_{L}} \left(\frac{U_{p}}{U_{\infty}}\right)^{2} \frac{1}{S_{t}} = \frac{(VR)^{2}}{4C_{L} S_{t}} \end{split}$$

Frequency corrected VR - 5

- Nagib et al (2006)
- Proposed to scale AFC effect on lift using
- Velocity ratio
- Strouhal number
- For large separated regions
- Low frequencies
- Problematic at $f \rightarrow 0 \dots$











- Velocity ratio (VR)
- Strouhal weighted VR (Nagib et al, 2006)
- Momentum coefficient
- Reynolds corrected C_{μ}
 - m=0.2 (Turb.), m=0.5 (Lam.)
- Vorticity flux coefficient

 $VR \equiv \frac{U_p}{U_e} \approx \frac{U_p}{U_{\infty}}$



$$C_{\mu} \equiv \frac{h}{c} (VR)^2$$

$$C_{\mu,\mathrm{Re}} \equiv \frac{h}{c} (VR)^2 \mathrm{Re}^m$$

$$C_{\Gamma} \equiv \frac{(VR)^2}{C_L S_t}$$



IAI-Pr8-40 Airfoil AFC Testing



- Chord 360mm, span 609mm
- Actuator at x/c=0.38
- Shallow angle, downstream excitation
- 14 segmented, individually controlled
- Piezo actuators (f_{Helmholtz}=1460Hz)
- Slots: 0.9mm by 40mm
- Power: 1.7w per actuator at 100vac









Ср



- Thick, Efficient, Low Re airfoil
- Mild, trailing edge stall
- Chord 360mm, span 609mm
- Actuator at x/c=0.38
- Shallow angle, downstream excitation
- 14 segmented, individually controlled
- Piezo actuators (f_{Helmholtz}=1460Hz)
- Slots: 0.9mm by 40mm











- Actuator at x/c=0.38
- Shallow angle, downstream excitation
- 14 segmented, individually controlled
- Piezo actuators (f_{Helmholtz}=1460Hz)
- Slots: 0.9mm by 40mm
- Two modes of operation:
- Pure sine, Anti-phase, F⁺>10
- AM, In-phase, F⁺~1
- Control of TBL separation
- Conducted amplitude scans at otherwise fixed conditions
- Scale lift increment vs amplitude







α=16°, Re=0.3M



• Velocity ratio (VR)







St corrected VR

α=16°, AM, F+=1





Momentum Coefficient







• Reynolds corrected Momentum Coefficient





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- m=0.2 (Turb.), m=0.5 (Lam.)
- used m=1...
- Vorticity flux coeff.

$$C_{\Gamma} \equiv \frac{(VR)^2}{C_L S_t} \qquad \qquad \mathbf{X} \qquad \mathbf{V}$$









- Three existing and two new excitation magnitude scaling options for active separation control at Reynolds numbers below one Million.
- The physical background for the scaling options was discussed and their relevance was evaluated using two different sets of experimental data.
- For *F*⁺~1, 2D excitation:
 - The traditional VR and C μ do not scale the data
 - Only the $\operatorname{Re}^*C\mu$ is valid
- This conclusion is also limited for positive lift increment.
- For F⁺>10, 3D excitation, the Re corrected Cμ, the St corrected velocity ratio and the vorticity flux coefficient, all scale the amplitudes equally well.
- Therefore, the Reynolds weighted $C\mu$ is the preferred choice, relevant to both excitation modes.
- Incidence also considered, using Ue from local Cp





Actuators also generate vorticity flux and alter circulation



- While the momentum coefficient
- For h/c = 0.005 (and $\theta/c = 0.005$)

$$C_{\mu} \equiv \frac{2h}{c} \left(\frac{U_p}{U_e}\right)^2$$

• So the ratio
$$C_{VF}/C_{\mu} \equiv \frac{0.01c}{h} / \frac{2h}{c} = \frac{0.005c^2}{h^2} \approx \frac{5x10^{-3}}{2.5x10^{-5}} = 200$$

Is very large; define Vorticity Flux Coefficient





AFC Open Forum: Actuators Comparison Criteria

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- Active control of boundary layer separation
- Actuation: key enabling technology
- Many actuator types exist
- No accepted criteria for actuator comparison
- Rare to find:
 - Energy, efficiency, weight, cost data







(Neuburger, '89, Re<100k)





- Actuator operates in still air
- F_a force generated
- When not/can not be measured could be estimated by
- With C from 0.25 to 0.5 (blowing only, velocity profile)
- U_p- peak generated velocity
- W_a actuator weight
- *P* actuator energy consumption
 - Electric, fluidic, combined

 $OFM \equiv$

 $F_a \approx C \rho A_a U_a^2$







 $AFM1 \equiv$



 $U_{\infty}L_{c}$

Aerodynamic Figure of Merit (1)

- Application, actuators and flow condition dependent
 - U_{∞} Free-stream velocity
 - L lift (baseline or controlled)
 - *D* Drag (baseline or controlled)
 - P actuator energy consumption
 - Situation: Have system and actuators
 - **Question:** Direct energy into power-plant or actuators?
 - **Answer:** Only when *AFM1* > 1 operate actuators





AFM-1 Example: 3D PZE



- Low Re Control of separation
- TAU Developed piezo-benders
- Considered energy efficiency
- Found that 3D excitation more effective than 2D









(Seifert et al, '99) 29







- Aerodynamic Figure of Merit (2) $U_{\infty}(L_{c} - W_{a}) / (U_{\infty}D_{c} + P)$ $AFM2 \equiv \frac{(U_{\infty}D_{c} + P)}{(L_{\infty})}$
- Application, actuators and flow condition dependent
 - U_{∞} Free-stream velocity
- *L* lift (baseline or controlled)
- D Drag (baseline or controlled)
- W_a weight of actuation system (incl. drivers, cables...)
- *P* actuation SYSTEM energy consumption
- Situation: Scaled AFM1>1 and actuation system weight known (or predicted)
- Question: Should we include actuation in system?
- Answer: only when AFM2 > 1 (probably 1.1 minimum)





(Not a tutorial nor a comprehensive review)

- External
 - Speakers on tunnel walls
 - Speakers in cavities
 - Mechanical rotary
 - Mechanical pneumatic
 - Ribbons + shakers
- Internal...









- Requirement: Sufficient control authority (...any type)
- Type (affecting method and feasibility of characterization)
- Mechanical: Amplitude, Mode shape...
- Micro balloons, micro flaps...



(Seifert et al., 1998, AIAA J., V. 36, N. 8.)



Premised Reactants

Figure 1. Schematic illustration of combustion-driven jet actuator

Microfabricated, Combustion-Driven Jet Actuators for Flow Control Applications

T. Crittenden and A. Glezer Woodruff School of Mechanical Engineering 801 Ferst Drive N.W. E. Birdsell and M. Allen School of Electrical and Computer Engineering 777 Atlantic Drive N.W. Georgia Institute of Technology, Atlanta, GA, USA 30332



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339

Active Flow Control by Surface Smooth Plasma Actua B. GÖKSEL, I. RECHENBERG, TUB











Actu power – 1000W/m Up=80m/s Wa>3kg



http://fdrc.iit.edu/research/docs/MAFC_XV_15_Briefing_Final.pdf





TAU Suction and Oscillatory Blowing (SaOB) Actuator







Background



Baseline

- Boundary layer separation control by steady suction (note date...)
- Steady, wall tangential blowing
- Oscillatory blowing (directed ZMF)
- The importance of flow instability for efficient AFC







- G. Arwatz (MSc), I. Fono, London MEMS IUTAM, Sep '06
- Available ZNMF devices limited to M=0.3
- Suction and oscillatory blowing high potential for efficient AFC
- Combination of steady suction and oscillatory blowing in one device
- High velocity, near sonic, output
- Wide range of output frequencies
- Enables 3D excitation
- Frequency proportional to flow rat
- No moving parts
- Low power consumption









• Combination of ejector and switching valve











• Overall Figure of Merit

$$OFM = \frac{F_a^2 U_p}{W_a P_a}$$

- Still air operation
- SaOB weight 15 gm
- Thrust-suction+blowing
- Note: OFM Log scale









- Actuator comparison criteria suggested
- Overall and Aerodynamic figures of merit
- New actuator for steady suction and oscillatory blowing was developed, modeled and characterized
- OFM for ZNMF, Plasma and SaOB actuators compared
- Rare to find data for AFM (Need: Weight, power, Up, L, D)
- Please measure and publish