A Piezoelectric Cryogenic Heat Switch

Amir E. Jahromi, 1,a and Dan F. Sullivan 1,b

¹NASA Goddard Space Flight Center, Cryogenics and Fluids branch code 552, Greenbelt, MD, U.S.A.

Abstract

We have measured the thermal conductance of a mechanical heat switch actuated by a piezoelectric positioner, the PZHS (PieZo electric Heat Switch), at cryogenic temperatures. The thermal conductance of the PZHS was measured between 4 K and 10 K, and on/off conductance ratios greater than 100 were achieved when the positioner applied its maximum force of 8 N. We discuss the advantages of using this system in cryogenic applications, and estimate the ultimate performance of an optimized PZHS.

Keywords: Heat switch, Thermal switch, mechanical heat switch, Piezoelectric heat switch, Adiabatic Demagnetization Refrigerator, liquid helium thermal switch

I. INTRODUCTION

Heat switches are used to control the transfer of heat between objects at different temperatures, and are particularly important in cryogenic environments. Applications in this context include minimizing the parasitic heat load from a cryocooler in a redundant refrigeration system [1], connecting instruments to and from a cryo-radiator [2], and transferring heat from a salt pill in an adiabatic demagnetization refrigerator (ADR) [3]. Many types of cryogenic heat switches exist, including gas gap, mechanically actuated, superconducting, and fluid-loop based [4]. In this work, we describe the construction and characterize the performance of a novel mechanical cryogenic heat switch actuated by a piezoelectric positioner, the PZHS.

Our motivation in developing the PZHS stems from the fact that many NASA science missions require cryogenic capabilities; consequently, heat switches often play a critical role in the success of these experiments. Since efficient photon detection tends to be greatly enhanced at lower temperatures $(T \le 1 \text{ K})$ [5], ADRs are firmly entrenched as NASA's preferred technology for detectors operating in this temperature regime. Current state of the art ADRs use multiple gas gap heat switches to transfer heat from the salt pill during the magnetization (heat rejection) stage of the refrigeration cycle. Although these heat switches have demonstrated the performance required, for example, by the Astro-H mission

a) Contact email: amir.e.jahromi@nasa.gov

b) Contact email: dan.f.sullivan@nasa.gov

[6], their use is limited to either very low (T < 0.3 K) or relatively high (T \sim 5 K) cryogenic temperatures. In addition, the hermetic joints required to contain the 3 He exchange gas within the switch housing have shown vulnerability to the mechanical stresses associated with thermal cycling and vibration. The PZHS is thus an attractive alternative technology to a gas gap heat switch, since this device has an essentially unlimited range of cryogenic operating temperatures, is mechanically robust, and is free from hermetic sealing requirements. In the remainder of this paper we will discuss the design and construction of the PZHS test apparatus, the measurements carried out to characterize the system, our experimental results, and plans for future work.

II. PZHS Design and Construction

The principle of operation of the PZHS is elegantly simple; our experimental apparatus for demonstrating this device is shown in Fig. 1. Two high-conductivity metallic plates act as independent thermal reservoirs, with one mounted to a support structure and the other fastened to the mobile stage of a piezoelectric positioner (an attocube ANPz101) [7]. When the positioner is energized with a series of positive voltage ramps, the lower plate moves upwards until mechanical contact is established with the upper plate. In this configuration, the switch is closed, and heat transfer occurs between the two plates. After the desired heat transfer is complete, energizing the positioner with negative voltage ramps moves it downwards until mechanical contact is lost between the plates, leaving the switch open.

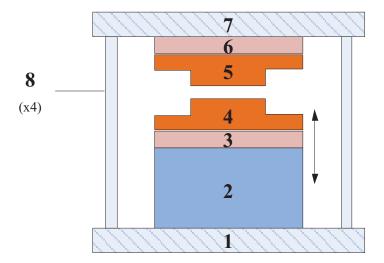


FIG. 1. A schematic of the PZHS design; 1 Base support plate 2 Piezoelectric positioner 3 Lower insulator 4 Lower conductor 5 Upper conductor 6 Upper insulator 7 Top support plate 8 Structure columns (x4)

The key figure of merit for this device is the *switching ratio*, *S*, defined by:

$$S \equiv \frac{k_c}{k_o} \tag{1}$$

where k_c and k_o are the thermal conductance of the PZHS when closed and opened, respectively. Proper engineering of a heat switch thus consists of maximizing k_c while minimizing k_o . When the switch is open, during steady state operation, parasitic heat flows from the high to low temperature reservoir via the structure columns. To mitigate this, and consequently minimize k_o , the columns were constructed from G-10 hollow rods, and the plates were mounted on Vespel SP1 insulators. To maximize k_c , the upper and lower plates were made from ultra high purity (99.999%) copper, with a contact area of 1.45 \times 1.45 cm².

With the expectation that the PZHS performance would be limited by the joint conductance, we paid particular attention to the preparation of the plate surfaces, striving to make them as flat and as parallel as possible. This would have the effect of increasing the effective PZHS contact area and thus enhancing k_c . Our method is shown in Fig. 2. The switch plates were bolted to a larger, hollowed out block, with walls slightly higher than the plate itself. This assembly was then placed on a lapping table, and the switch surfaces were sanded by pouring isopropanol alcohol on sand paper of progressively finer paper grits, culminating in the use of a special cotton impregnated with metal polish. Once the surfaces were immaculate, the switch plates were cleaned and gold plated with $\sim 1~\mu m$ of Au. The plating prevents tarnishing of the copper, and also acts as a "cushion" to the switch surfaces, further enhancing the effective contact area. Table 1 summarizes the properties of the main components of the PZHS.

Table 1 Properties and dimensions of major components of our PZHS

Parts	Material	Dimension
Upper and Lower conductor	99.999% pure copper	Contact area square of 1.45 cm on each side
Upper and Lower insulator	Vespel SP-1	7 mm thick square block
Support columns	G-10	4.3 cm long, 6.3 mm OD, 3.2 mm ID

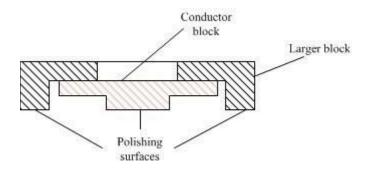
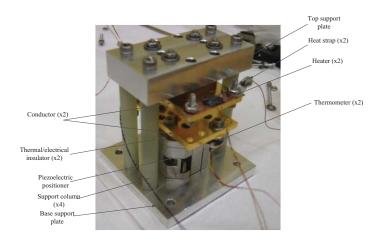


FIG 2. Lapping method for the copper plates to ensure flat, parallel and smooth PZHS mating surfaces

In order to establish the desired temperature of the reservoirs independently, a small thermometer and heater were mounted on each plate, and the temperatures were maintained using a PID controller. To allow for reasonable thermal time constants (\sim minutes), thin *electrically insulated* copper wires were attached from each reservoir to the cold stage of the test cryostat, a cryocooler with a base temperature of \sim 3 K. Contact between the plates was checked using a two-wire resistance measurement. With the switch open, no continuity was observed; with the switch closed, continuity was confirmed, with a measured resistance of \sim 80 Ω . We note that all of our closed switch data was obtained with the positioner applying its maximum specified force of 8N. A photograph of the complete PZHS assembly is shown in Fig. 3.



III. Experiments and results

Figure 4 shows a simplified schematic of the conductance network in our PZHS test apparatus. The upper switch plate, lower switch plate, and cryocooler cold plate temperatures are labeled T_U , T_L , and T_b , respectively. The conductance of the mechanical joint is k_c , while the upper switch plate is connected to the cold plate through a conductance k_U , and the lower switch plate is connected to the cold plate through a conductance k_L . With the switch open, $k_c = 0$, and k_U and k_L can be determined as a function of temperature by setting T_U and T_L equal:

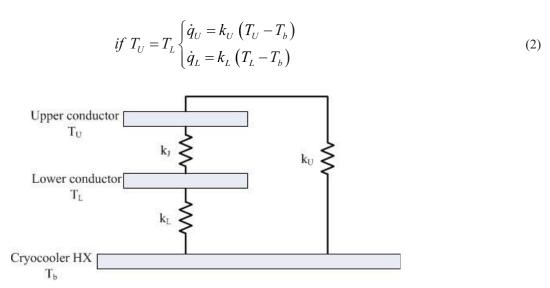


Figure 4. A simplified conductance network of the PZHS test apparatus.

We observed that both k_U and k_L were linearly dependent on reservoir temperature, with best-fit results (in mW/K) of $k_U = 3.2 \text{x} 10^{-6} + (6.4 \text{x} 10^{-6}) T_U$ and $k_L = 5.4 \text{x} 10^{-6} + (4.3 \text{x} 10^{-6}) T_L$. After measuring the conductance from the reservoirs to the cold plate, the joint conductance k_c was determined by closing the switch, independently controlling the temperature of each plate, and applying the following equations:

$$if T_{U} = T_{L} + \delta T \begin{cases} \dot{q}_{U} = k_{U} \left(T_{U} - T_{b} \right) + k_{J} \left(T_{U} - T_{L} \right) \\ \dot{q}_{L} = k_{L} \left(T_{L} - T_{b} \right) \end{cases}$$

$$if T_{U} + \delta T = T_{L} \begin{cases} \dot{q}_{U} = k_{U} \left(T_{U} - T_{b} \right) \\ \dot{q}_{L} = k_{L} \left(T_{L} - T_{b} \right) + k_{J} \left(T_{L} - T_{U} \right) \end{cases}$$

$$(3)$$

Solving the set of equations in Eq. (3) yields the conductance of the mechanical joint, k_c . We plot the results of our experiments in Fig. 5 for temperatures from 4 K to 10 K. A conservative estimate of the PZHS switching ratio at 4 K can be obtained by setting $k_o = k_U \sim 0.028$ mW/K, which yields S ~ 100 . We note that the thermal conductance for a solid piece of copper at 4 K with the same dimensions as our switch plates is 62 W/K, which represents the theoretical upper limit of performance for the PZHS. A more realistic estimate, based on studies of bolted mechanical joints (Cryogenics 44(5): 293-299 (2004)), suggests a limit of ~ 10 W/K. Thus, as anticipated, the PZHS conductance is limited by k_c , and future work should focus on methods to improve this parameter.

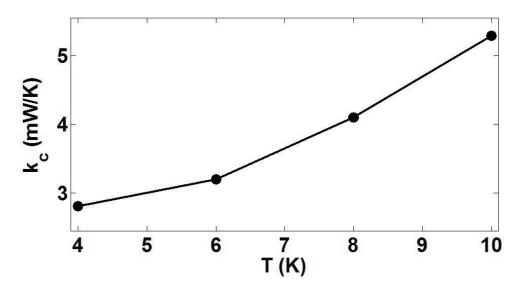


FIG 5. Thermal conductance of the PZHS vs. temperature.

IV. Conclusions

We have demonstrated a mechanical heat switch, the PZHS, actuated by piezoelectric elements for use in cryogenic applications, with a switching ratio $S \sim 100$ at 4 K. In principle, such a heat switch would be sufficiently versatile to operate at all cryogenic temperatures of interest to NASA. The PZHS

has the potential for significant improvement by using piezoelectric positioners with larger clamping forces, in addition to spring loading on a thermal reservoir. We hope to incorporate these and other improvements in a subsequent version of the PZHS.

REFERENCES

¹Van Oost, S., Bekaert, G., Bhatti Sabca, R.S., Scull, S., and Jewell, C., "A Heat Switch for Space Cryocooler Applications," Proceedings of the 4th European Symposium on Space Environmental and Control Systems, Florence, Italy, 21-24 October, 1991 (ESA SP-324.December 1991).

²Bugby, D.C., Cepeda-Rizo, J., and Rodriguez, J. I., "Thermal switching Cryogenic Heat Pipe," Proceedings of the 16th International Cryocoolers Conference, Boulder, CO, 2011.

³Shirron, Peter J., et al. "Design of a 3-stage ADR for the soft x-ray spectrometer instrument on the ASTRO-H mission." *SPIE Astronomical Telescopes and Instrumentation: Observational Frontiers of Astronomy for the New Decade*. International Society for Optics and Photonics, 2010.

⁴M. Donabedian, Spacecraft Thermal Control Handbook Vol. II: Cryogenics. El Segundo, CA; 2003.

⁵Glass, I. S. (1999). Handbook of Infrared Astronomy. New York: Cambridge University Press. ISBN 0-521-63311-7.

⁶M. O. Kimball. P. J. Shirron, AIP Conf. Proc. 1434, 853 (2012); doi 10.1063/1.4707000.

⁷Attocube Systems AG., Königinstrasse 11a. 80539, München. Germany.