

# Launch Load Resistant Sp. Mechanism Bearings Made F om Ni-Ti Superelastic Intermetallic N

Christopher DellaCorte NASA, Glenn Research Center And Lewis (Chip) E. Moore III NASA Marshall Space Flight Cente

42<sup>nd</sup> Aerospace Mechanisms Symposium May 14th, 2014 Baltimore, Maryland

# ecraft iterials

brought to you by TCORE

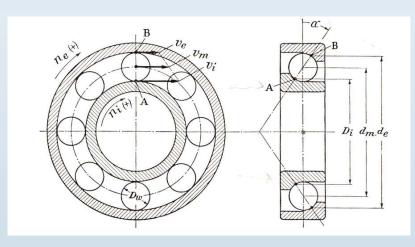


### Motivation: the ride into space can be rough

(Vibration/loads impact bearings and components)

- Bearing and component materials must be:
  - Hard (Rockwell C58 or better)
  - Wear-resistant and compatible with existing lubricants
  - Resistant to rolling contact fatigue (RCF)
  - Fracture resistant
  - Corrosion resistant (preferably immune)
  - Low density (to reduce centripetal loads at high rpm)
  - Capable of producing ultra-smooth surface finishes
  - Dimensionally stable and easy to manufacture



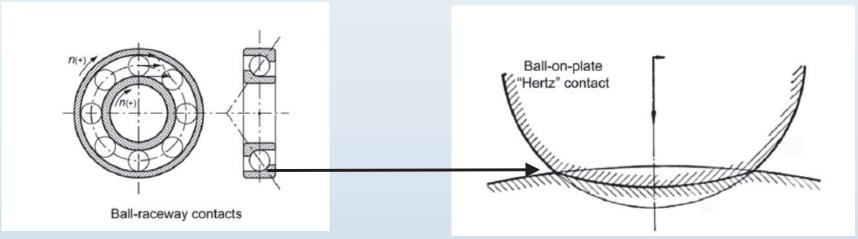




# Contact Engineering: Ball-Race

#### When hard surfaces contact

- Forces are transmitted at small, concentrated contact points (Hertz).
- Resulting stresses cause deformations that help "spread the load".
- Contact area is a function of the geometry, material stiffness and load.
- High stiffness (modulus) inhibits deformations leading to small contact area and high stresses (contrast with a tire contacting the ground).



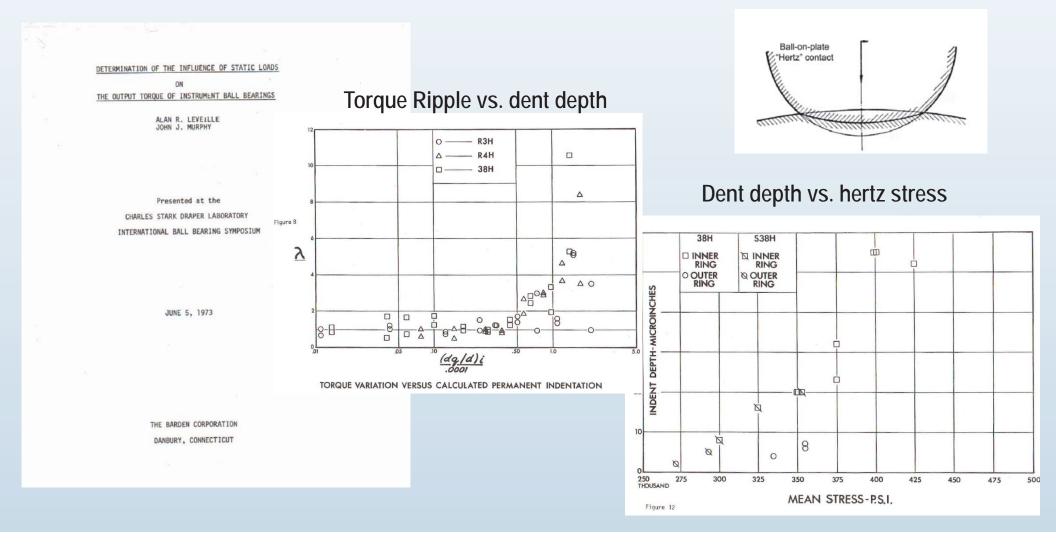
#### Hard surfaces can dent

- Even modest loads can exceed stress capability limits.
- Bearing raceways are particularly prone to Brinell dent damage.



# Ball-Race Static Load Capacity: Leveille & Murphy (Dent depth vs. running torque noise)

- Classic 1973 paper on dent depth/ball diameter (dp/D) effects
  - Showed that dp/D~0.0001 criterion too aggressive for precision bearings with respect to torque ripple. Proposed dp/D~0.00003 to 0.00005

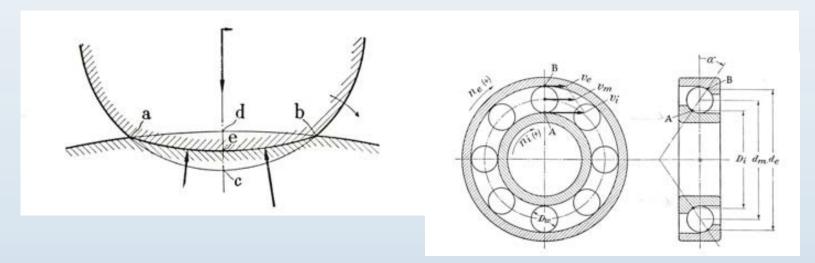




## Contact Engineering: Geometry, Loads and Materials

#### Engineering Mitigations

- Reduce loads through vibration isolation.
- Reduce the stresses through margin additions such as increased bearing size and increased ball-race conformity.
- Use harder materials less prone to denting.



#### Implications

- Load reduction and vibration isolation can add mass.
- Bearing design and material changes introduce other complications.



## Bearing Material: State-of-Art (SOA)

(Current suite of candidates is severely limited)

- Four general types of bearing materials:
  - Steels (Corrosion resistant steels, martensitic, austenitic)
  - Ceramics (Si<sub>3</sub>N<sub>4</sub> balls + steel races, a.k.a., hybrid bearings)
  - Superalloys (e.g., jet turbine blade alloys)
  - Non-ferrous alloys (bronze, nylon etc.)
- Each of these has inherent shortcomings:
  - Hard steels are prone to rusting (even "stainless steels" like 440C)
  - Superalloys and austenitic stainless steels (304ss) are soft.
  - Ceramics have thermal expansion mismatch and dent steel races
  - Non-Ferrous materials are weak and lack temperature capabilities
- No known bearing material blends all the desired attributes:
  - High hardness, corrosion immunity, toughness, surface finish, electrical conductivity, non-magnetic, manufacturability, etc.



## **Technical Opportunity:**

60NiTi (a.k.a. NiTiNOL 60)

#### 60NiTi Basics:

- Invented by W.J. Buehler (late 1950's) at the Naval Ordinance Laboratory (NiTiNOL stands for Nickel-Titanium Naval Ordinance Lab).
- Contains 60 wt% Nickel and 40 wt% Titanium
- 60NiTi is not a metal or a ceramic: a weakly ordered inter-metallic compound.
- A member of the super-elastic family. It is dimensionally stable.
- 60NiTi can be hardened to Rc 60+.
- Its cousin (55NiTi), the widely used shape memory alloy, is soft and dimensionally unstable.
- 60NiTi recognized by Buehler for bearings but too difficult to manufacture.
- Modern (ceramic) processing methods now enable 60NiTi bearings with remarkable properties.



# **Technical Properties Comparison:**

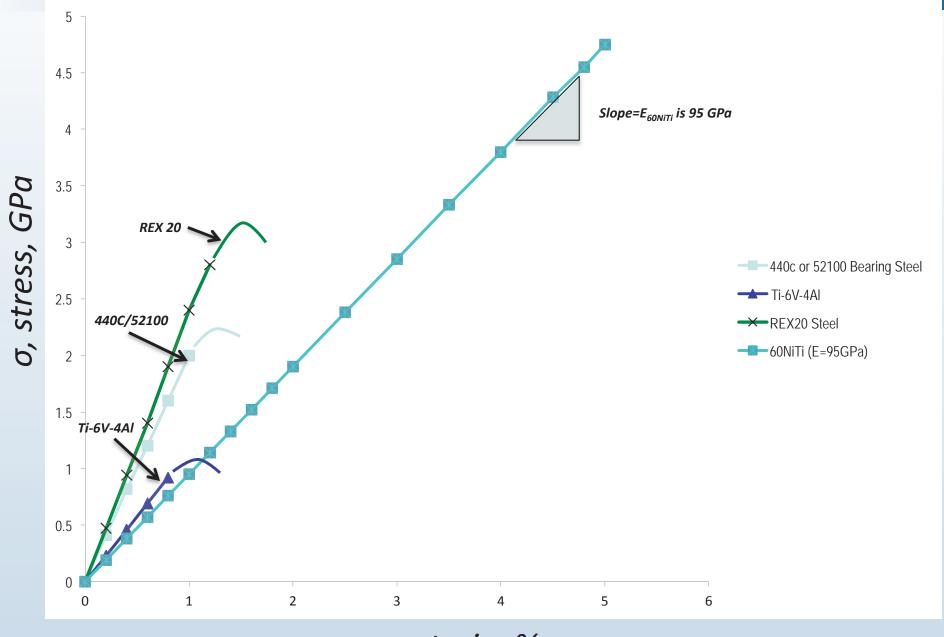
Property	60NiTi	440C	$Si_3N_4$	M-50
Density	6.7 g/cc	7.7 g/cc	3.2 g/cc	8.0 g/cc
Hardness	56 to 62 HRC	58 to 62 HRC	1300 to 1500 Hv	60 to 65 HRC
Thermal conductivity	~9 to 14	24	33	~36
W/m-°K				
Thermal expansion	~11.2×10 <sup>-6</sup> /°C	10×10 <sup>-6</sup> /°C	2.6×10 <sup>-6</sup> /°C	~11×10 <sup>-6</sup> /°C
Magnetic	Non	Magnetic	Non	Magnetic
Corrosion resistance	Excellent	Marginal	Excellent	Poor
	(Aqueous and			
	acidic)			
Tensile/(Flexural	~1000(1500) MPa	1900 MPa	(600 to 1200) MPa	2500 MPa
strength)				
Young's Modulus	~95 GPa	200 GPa	310 GPa	210 GPa
Poisson's ratio	~0.34	0.3	0.27	0.30
Fracture toughness	~20 MPa/√m	22 MPa/√m	5 to 7 MPa/√m	20 to 23 MPa/√m
Maximum use temp	~400 °C	~400 °C	~1100 °C	~400 °C
Electrical resistivity	~1.04×10 <sup>-6</sup> Ω-m	~0.60×10 <sup>-6</sup> Ω-m	Insulator	~0.18×10 <sup>−6</sup> Ω-m

#### Primary Points

- Modulus is ½ that of steel, yet hardness is comparable.
- Tensile strength akin to ceramics.



## 60NiTi: Stress-Strain Behavior



ε, strain, %



## Brinell Test: Elucidates static load capacity

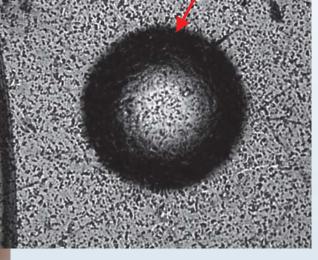
How well does 60NiTi resist dents?

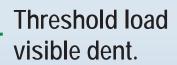
Deep Brinell dent.

- Brinell number

dp/D vs. stress (Leveille)



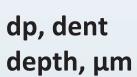


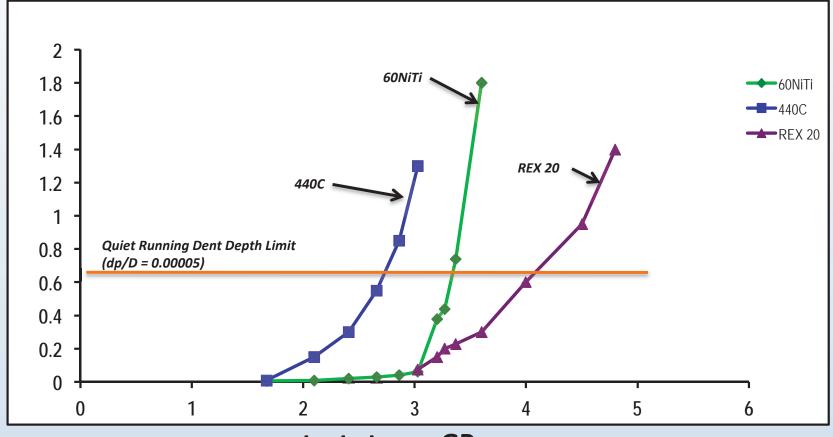




### Dent Depth vs. Hertz Contact Stress

(12.7 mm diameter Si<sub>3</sub>N<sub>4</sub> ball against 60NiTi plate)



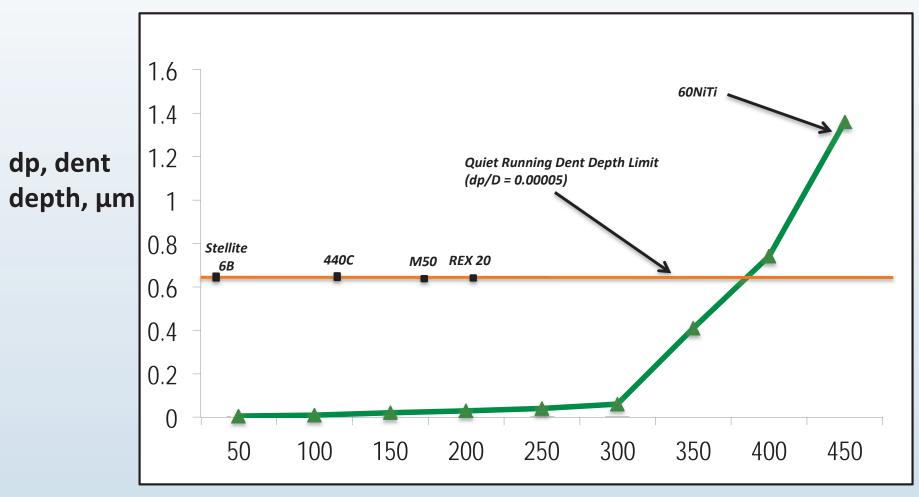


 $\sigma_{avg}$ , contact stress, GPa



#### Dent Depth vs. Load

(12.7 mm diameter Si<sub>3</sub>N<sub>4</sub> ball against 60NiTi plate)

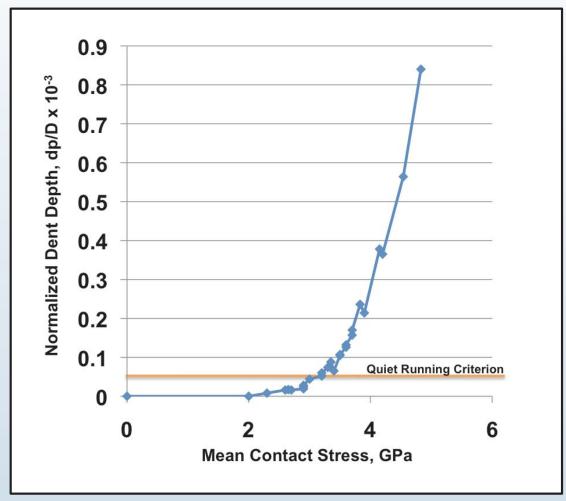


W, indentation load, Kgf



#### **Effects of Indenter Diameter**

Dent Depth vs. Stress: 6.4 to 12.7mm Si<sub>3</sub>N<sub>4</sub> indenter balls



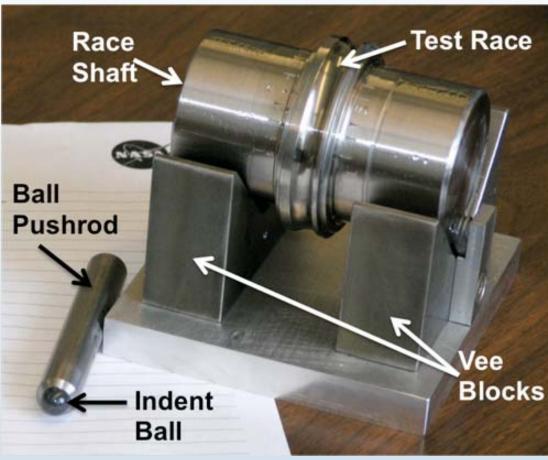
- Implications
  - Hertz stress relations work well for hard balls against flat plates.



## Dent Depth vs. Stress: On bearing races?

50mm bore 60NiTi-Hybrid Bearing specimens





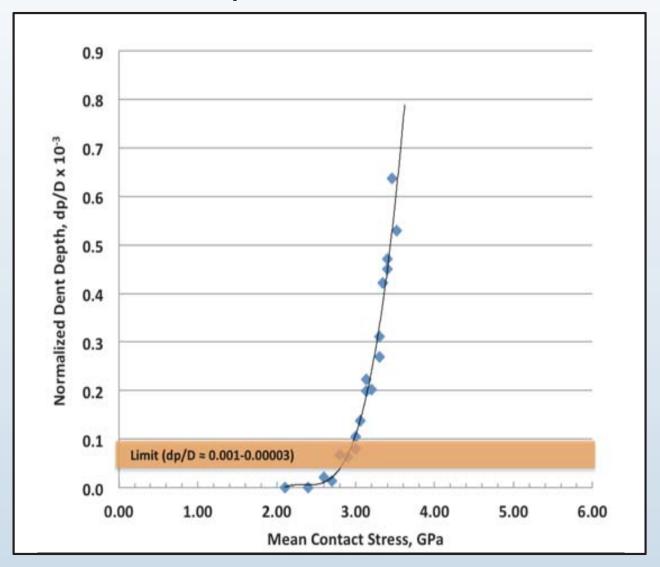


Full scale (50mm bore) bearing inner race. Dented with 8.74mm Si<sub>3</sub>N<sub>4</sub> ball.



## Dent Depth vs. Stress: Data Plot

#### Normalized Dent Depth Versus Mean Hertz Contact Stress

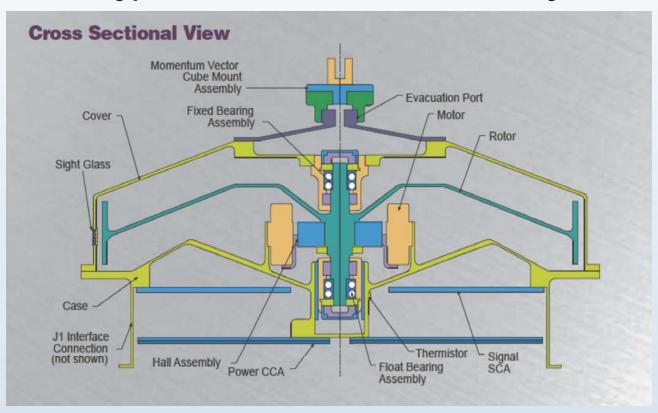


Exemplary dent resistance applies to real bearing races as well as flat plates.



## **Notional Bearing Application**

#### **Typical Reaction Wheel Assembly**



- -Based on Honeywell Corporation Model HR 0610 design.
- -5 kg wheel supported on four R4 ball bearings.

How might NiTi bearings compare to steel w.r.t. static load capacity?



# Single Ball on Race Calculations

#### Reaction Wheel Assembly Bearing Configurations Assessed

[Ball: 8.74 mm dia., Inner Race Curvature Radii: ball-path, 4.25 mm; cross-race, 1.27 mm.]

Configuration	Ball material	Race material	Limiting contact	Single ball-race load
no.			Stress, <sup>a</sup> GPa (ksi)	limit, N (lb)
I	440C	440C	2.5 (350)	196 (44)
II	$Si_3N_4$	440C	2.5 (350)	138 (31)
III	60NiTi	440C	2.5 (350)	463 (104)
IV	60NiTi	60NiTi	2.5 (350)	846 (190)
V	$Si_3N_4$	60NiTi	2.5 (350)	374 (84)
VI	60NiTi	60NiTi	3.1 (450)	1780 (400) <sup>c</sup>
VII	$Si_3N_4$	60NiTi	3.1 (450)	801 (180)
VIII	⁵REX20	REX20	3.8 (550)	587 (132)
IX	$Si_3N_4$	REX20	3.8 (550)	467 (105)

<sup>&</sup>lt;sup>a</sup>Mean Hertz contact stress.

<sup>&</sup>lt;sup>b</sup>REX20 properties: Young's Modulus (E): 234 GPa; Poisson's Ratio (v): 0.30.

<sup>&</sup>lt;sup>c</sup>Hertz calculations may be invalid due to excessively deformed geometry.



# Calculated Shaft Load Capacityb

#### Reaction Wheel Assembly Bearing Configurations Assessed

[Ball: 8.74 mm dia., Inner Race Curvature Radii: ball-path, 4.25 mm; cross-race, 1.27 mm.]

Configuration	Ball material	Race	Shaft load capacity,	RWA load
Case		material	kN (lb)	capacity, g
I	440C	440C	1.4 (316)	28.6
II	$Si_3N_4$	440C	1.0 (223)	20.2
III	60NiTi	440C	3.3 (748)	67.9
VI	60NiTi	60NiTi	a12.8 (2880)	<sup>a</sup> 261.2
VII	Si <sub>3</sub> N <sub>4</sub>	60NiTi	5.8 (1296)	118
VIII	REX20	REX20	4.2 (950)	86.2
IX	Si <sub>3</sub> N <sub>4</sub>	REX20	3.4 (756)	68.5

<sup>&</sup>lt;sup>a</sup>Hertz calculations may be invalid due to excessively deformed geometry

- -Si<sub>3</sub>N<sub>4</sub> ball-60NiTi race offers 2x (vs. Rex20) to 5x (440C) improvement.
- -60NiTi ball-60NiTi race amplifies load capacity effect.
- -Additional load capacity may other designs such as smaller or fewer bearings.

<sup>&</sup>lt;sup>b</sup>Shaft supported by four R4 bearings. Bearing radial load capacity estimated using Derner & Pfaffenburger (9/5) load sharing distribution model.

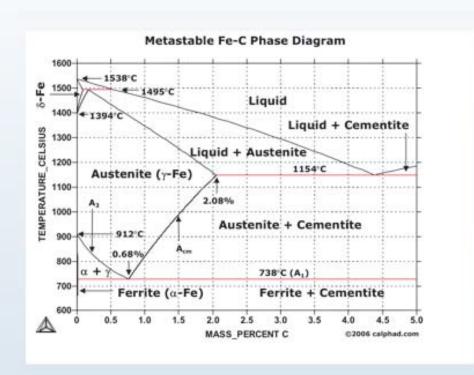


## **Status and Summary Remarks**

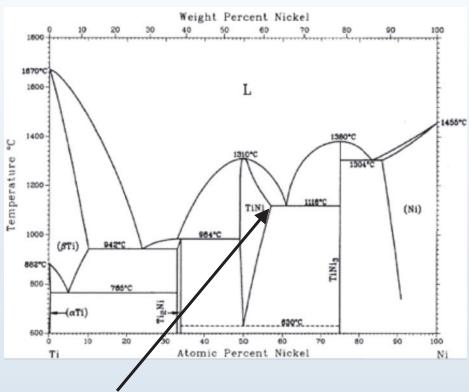
- •60NiTi has been successfully fabricated into precision bearing balls and races.
- •60NiTi is hard yet has a low elastic modulus and large elastic deformation range enabling high static load capacity.
- •Combination of aqueous corrosion immunity, non-magnetic and electrical conductivity not found in any other hard bearing material.
- •Low modulus and high elasticity of superelastic gives it more load capacity than that inferred from hardness alone.
- •Under load, the reduced modulus may allow better load sharing amongst rolling elements, further reducing local stresses thereby increasing bearing load capacity.
- •As the technology matures, more improvements and applications will emerge.



# Closing Thoughts: Materials Design Space



Fe-C system has yielded literally thousands of alloys and variants following centuries of development.



NiTi explorations to date have been limited to very narrow region.

Though much more R&D remains to commercialize 60NiTi and other superelastic intermetallic materials for use in bearings, gears and other mechanical systems, early indications are very promising.



## Thank You!