

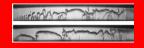
DEVELOPMENT AND CAPABILITIES OF ISS FLOW BOILING AND CONDENSATION EXPERIMENT

Henry Nahra¹, Mohammad Hasan¹, R. Balasubramaniam¹, Michelle Patania¹, Nancy Hall¹, James Wagner¹, Jeff Mackey², Bruce Frankenfield¹, Daniel Hauser¹, George Harpster¹, David Nawrocki¹, Randy Clapper¹, John Kolacz¹, Robert Butcher¹, Rochelle May¹, David Chao¹, Issam Mudawar³, Chirag R. Kharagante³, Lucas E. O'Neill³

¹NASA Glenn Research Center, 21000 Brookpark Rd., Cleveland, OH 44135

²Vantage Partners LLC, 3000 Aerospace Parkway, Brookpark 44142
³ Purdue University Boiling and Two-Phase Flow Laboratory (PU-BTFPL), 585 Purdue Mall, West Lafayette, IN47907, U.S.A

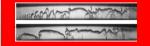




Agenda

- Experiment Objective
- Top Level Science Requirements and Constraints
 - ISS Constraints
 - Mass, Volume, Power, Cooling Constraints
- Test Sections
- Fluid System
 - Engineering Schematic
 - Fluid System nPFH Module
 - Fluid System Cooling Module
 - Fluid System Preheater
- Breadboard Testing
 - Pressure data and Inlet conditions for Condensation Experiment
 - Pressure data and Inlet conditions for Flow Boiling Experiment
 - Video Imaging and Capabilities
- Future Work





FBCE Science Objectives

The proposed research aims to develop an integrated two-phase flow boiling/condensation facility for the International Space Station (ISS) to serve as primary platform for obtaining two-phase flow and heat transfer data in microgravity.

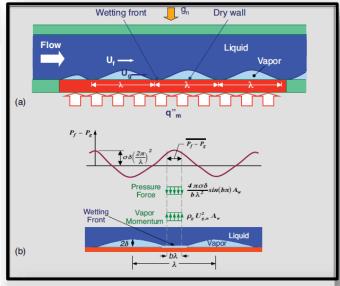


Key objectives are:

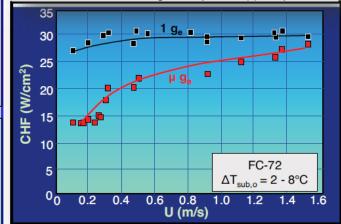
- Obtain flow boiling database in long-duration microgravity environment
- Obtain flow condensation database in long-duration microgravity environment
- Develop experimentally validated, mechanistic model for microgravity flow boiling critical heat flux (CHF) and dimensionless criteria to predict minimum flow velocity required to ensure gravity-independent CHF
- 4. Develop experimentally validated, mechanistic model for microgravity annular condensation and dimensionless criteria to predict minimum flow velocity required to ensure gravityindependent annular condensation; also develop correlations for other condensation regimes in microgravity

Applications include:

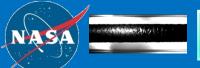
- Rankine Cycle Power Conversion System for Space
- Two Phase Flow Thermal Control Systems and Advanced Life Support Systems
- 3. Gravity Insensitive Vapor Compression Heat Pump for Future Space Vehicles and Planetary Bases
- 4. Cryogenic Liquid Storage and Transfer

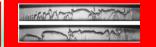


Interfacial Lift-off Model: (a) schematic representation of wavy vapor layer. (b) Balance of vapor momentum and interfacial pressure difference at moment of wetting front separation.

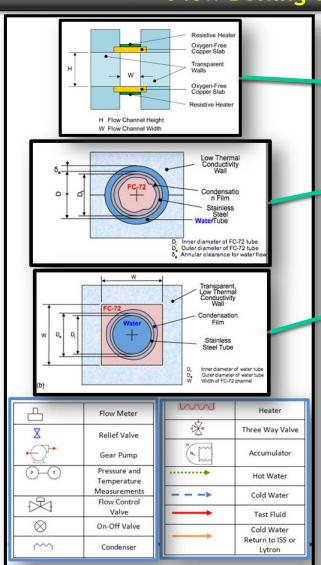


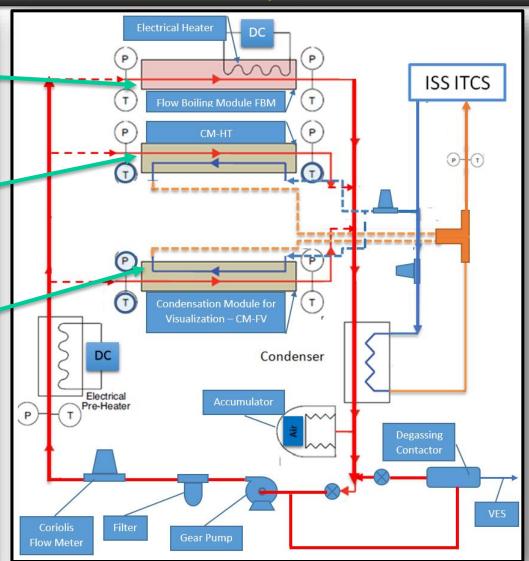
- •Science Requirements Document for FBCE, March, 2013
- •Science Concept Review Presentation, December 2011



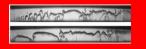


Flow Boiling and Condensation Fluid Systems



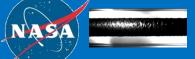


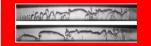




Top Level Science Requirements and Constraints

- Requirements-Fluid System
 - Deliver flow rates between 2 and 14 g/s of nPFH for Condensation Experiments and 2 to 40 g/s for Flow Boiling Experiments
 - Deliver the required power up to 1660 W to the fluid
 - Deliver the required system pressure of 100 and 150 kPa
 - Volume increase is accommodated with an accumulator
 - Deliver the required thermodynamic conditions of the fluid at the entrance of the test modules (subcooled, saturated and two-phase mixture)
 - Provide the fluid cooling function
- Constraints
 - Limitation on the available power (~1660 W total available for heating) and available heat dissipation (~1600 W)
 - ITCS cooling water flow rate up ~50 g/s to and returning stream temperature requirement of 40-49 °C
 - Volume constraint 91.44x121.92x48.28 cm³ (36x48x19 in³)
 - Mass constraint (~200 kg max)



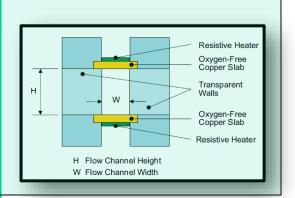


Flow Boiling and Condensation Test Modules



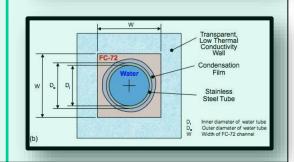
Flow Boiling Module Assembled

- Flow Boiling Module
 - Subcooled, saturated and 2-phase Inlet condition at:
 - Mass Flow Rate 2.5 to 40 g/s
 - Heat Flux < 60 W/cm²



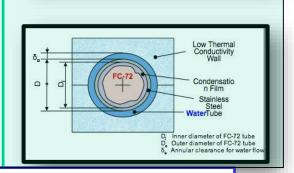


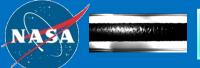
- Condensation ModuleFlow Visualization
 - Saturated vapor Inlet condition
 - Mass Flow Rate 2 to 14 g/s



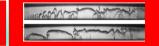


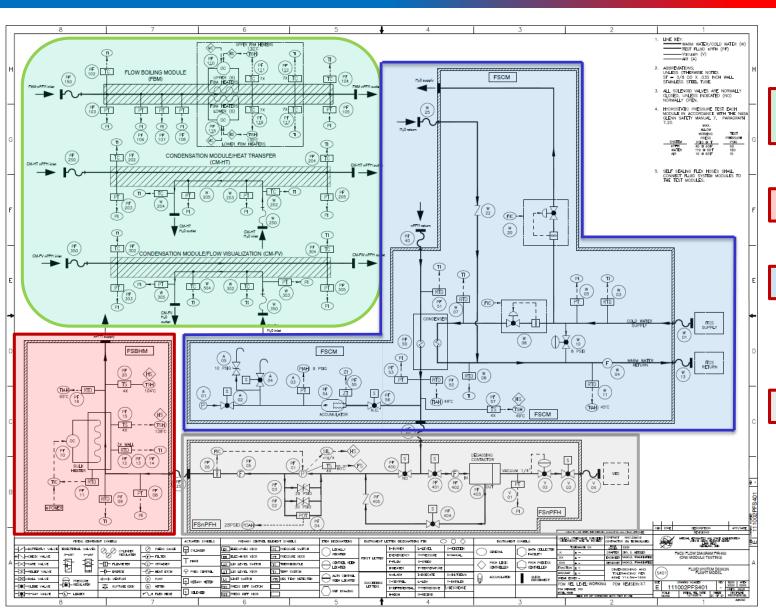
- Condensation ModuleHeat Transfer
 - Saturated vapor Inlet condition
 - Mass Flow Rate 2 to 14 g/s





FBCE Fluid System Architecture



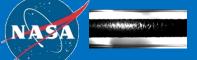


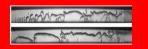
Test Modules

FS-BHM

FS-CM

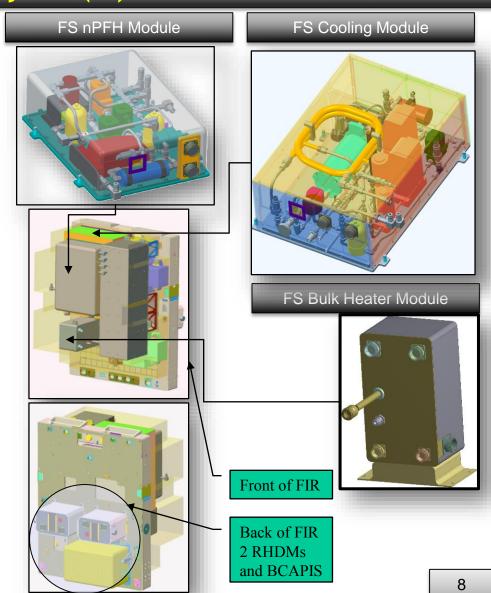
FS-nPFHM

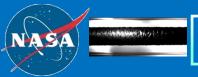




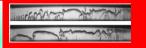
FBCE Fluid System (FS) Modules

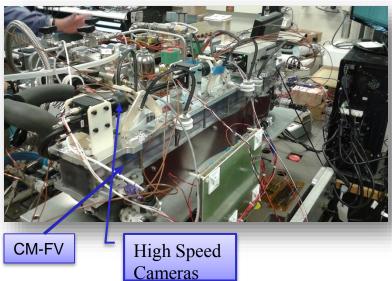
- Fluid System (FS) nPFH Module
 - Consists of:
 - Pump
 - Filter
 - Coriolis flow meter
 - Degassing
- FS Cooling Module
 - Consists of:
 - Condenser
 - Accumulator
 - Coriolis flow meters
- FS Preheater Module
 - Consists of:
 - Preheater
 - Electronics and Control





Breadboard/Brassboard Model

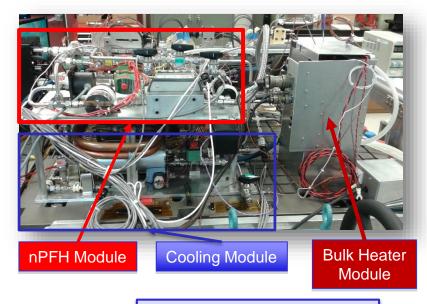


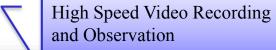




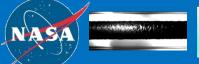
Data Acquisition

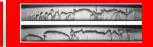
and Control







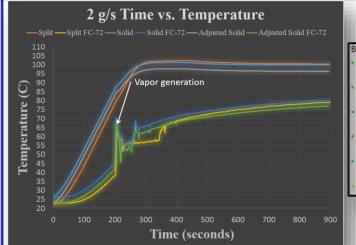


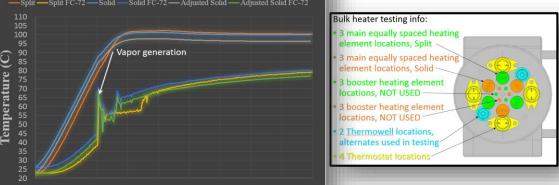


Experiments...Heater Performance Study

Similar Conditions used for Split and Solid Sheath:

- Low Flow Test (2 g/s), 100 kPa
 - Average FC-72 into Bulk Heater:
 - 1.98± 0.11 g/s, 23.6± 0.9 C
 - Average Water into Condenser:
 - 20 ± 0.1 g/s, 20.4 ± 0.1 C
 - Average Water into CM-FV:
 - 5.07 ± 0.01 g/s, 21.2 ± 0.2 C
- High Flow Test (40 g/s), 100 kPa
 - Average FC-72 into Bulk Heater:
 - 40 ± 0.2 g/s, 32.76 ± 0.2 C
 - Average Water into Condenser:
 - 15.04 (\pm 0.02) g/s, 20.4 (\pm 0.1) C
 - Average Water into CM-FV:
 - 15.1 (± 0.1) g/s, 20.4 (± 0.1) C



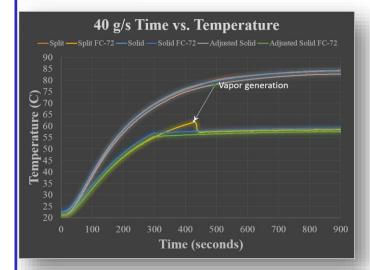


Solid sheath heating elements seem to heat quicker

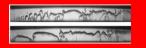
- Cartridge heaters pressed into the aluminum holes (typical installation) should provide even quicker héatina
- Pressed in cartridge heaters are less likely to fall out

The solid sheath cartridge heaters were run at a lower voltage than the split sheath cartridge heaters

- Electrical tests indicated that the solid sheath had lower electrical resistance than the split sheath cartridge heaters
- Even with less voltage, the solid sheath heated quicker



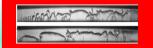




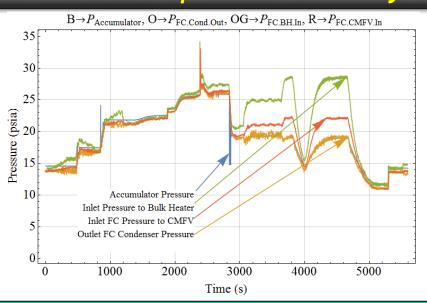
Experiments...Heater Performance Study

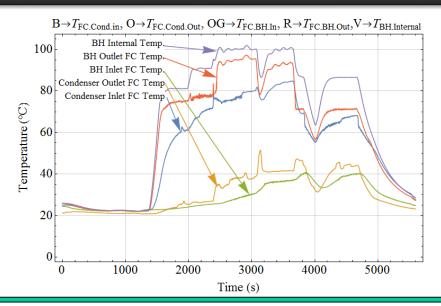
- Solid sheath cartridge heaters
 - Faster heating, better fit and lower contact resistance, operates at lower voltage
 - Bulk heater manufacturer makes solid sheath cartridge heaters
 - Could have the manufacturer build this as COTS
- Split sheath cartridge heaters
 - Typically used for ease of removal to facilitate replacement
 - FBCE does not require this feature
 - Allows a TC in the hole with the cartridge heaters
 - Bulk heater manufacturer does not make solid sheath cartridge heaters
 - GRC assembly of cartridge heaters into bulk heater

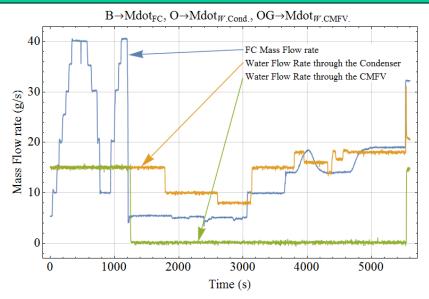


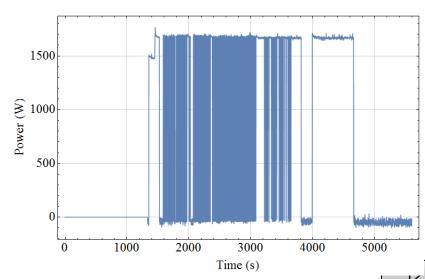


Experiments...Fluid system Performance Assessment

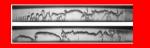






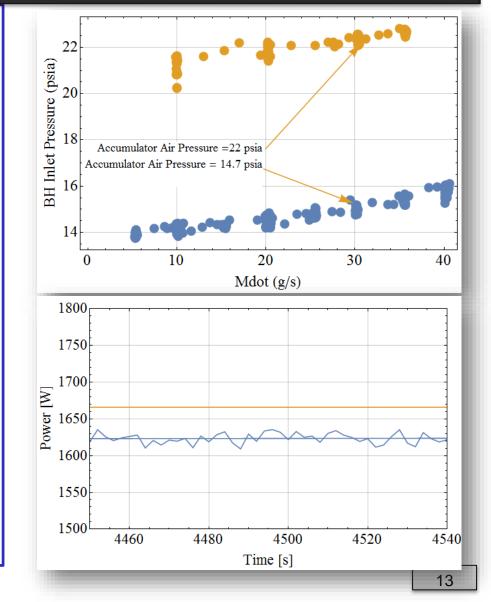


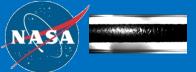


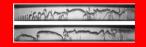


Experiments...Fluid system Performance Assessment

- System pressure increases steadily with flow rates
- Heater power averaged over periods of time where FC-72 flow rate is constant
- Heat gained by FC-72 in bulk heater was calculated from thermodynamic measurements
- An average of 54 W lost from heater at 14 g/s of FC and slightly lower heat loss of 45 W at 10 g/s
- Similar analysis on CMFV showed a loss of 65 W to ambient

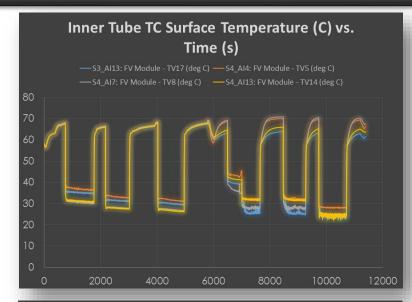


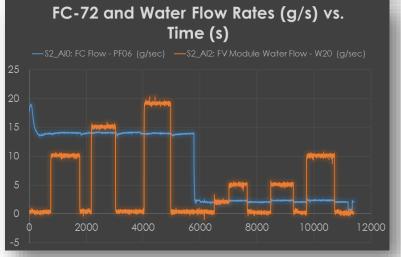


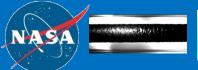


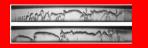
Experiments...Transients in Flow Condensation

- Flow loop along with selected test modules is capable of obtaining full transient to steady state data along with high speed imaging
- Example of Transient Test Run with CM-FV
 - FC-72 vapor with specified inlet conditions (flow rate and temperature) introduced into the condensation module with no cooling water flow and allowed sufficient time for steady state
 - Cooling water flow rate started simultaneously with high speed video recording
 - High speed video recorded at 1000 fps for 29 seconds to capture the condensation transient
 - Relevant data (water and surface temperature, FC-72 and water flow rates, pressure) continuously recorded at 1 Hz
 - At the end of 10 minutes time interval, two seconds of high speed video is recorded to provide a comparison between the steady state and the transient

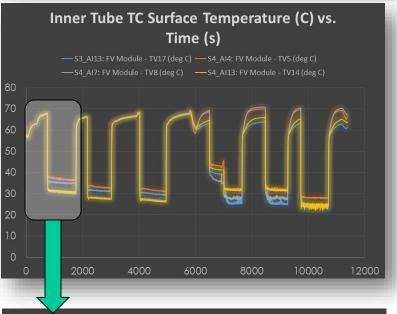


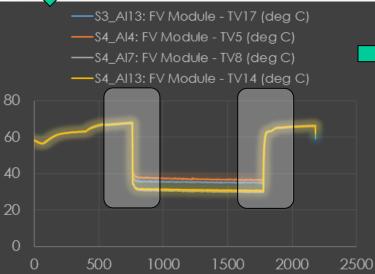




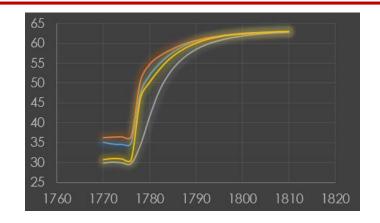


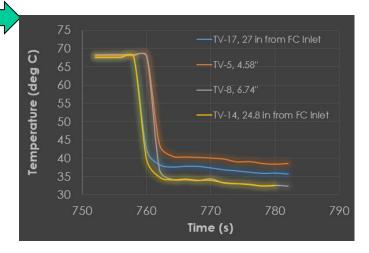
Experiments...Transients in Flow Condensation

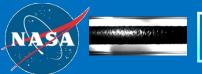


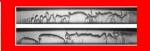


- ☐ Transient of 10 seconds observed upon turning on water flow rate
- □ A time scale of about 25 second observed upon turning off cooling water to CMFV



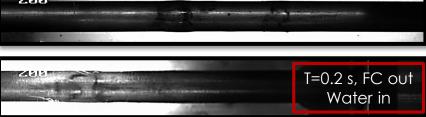


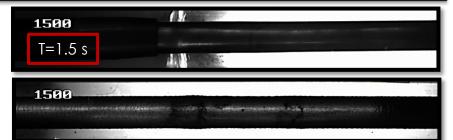




Experiments...Transients in Flow Condensation

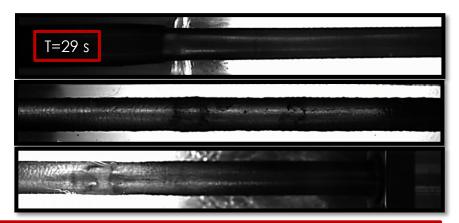


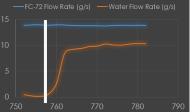




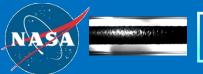


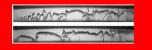






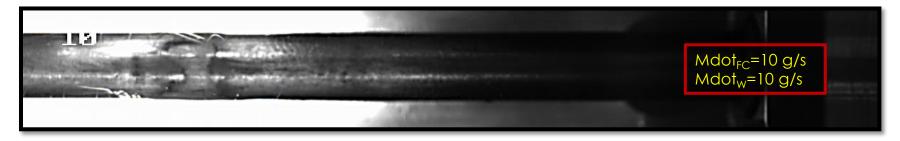
- ☐ High speed imaging (1000 fps) of condensation in CM-FV in the horizontal configuration for 0.2 s, 1.5 s, 10 s and 29 s.
- Imaging started simultaneously with water flow rate as designated by the white vertical line in the graph on the left

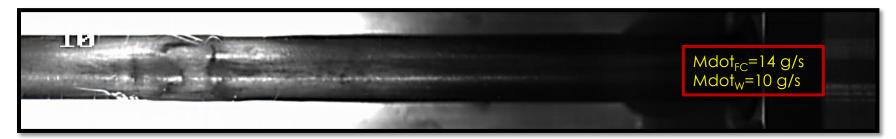




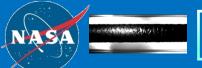
Experiments...Transients in Flow Condensation

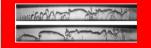




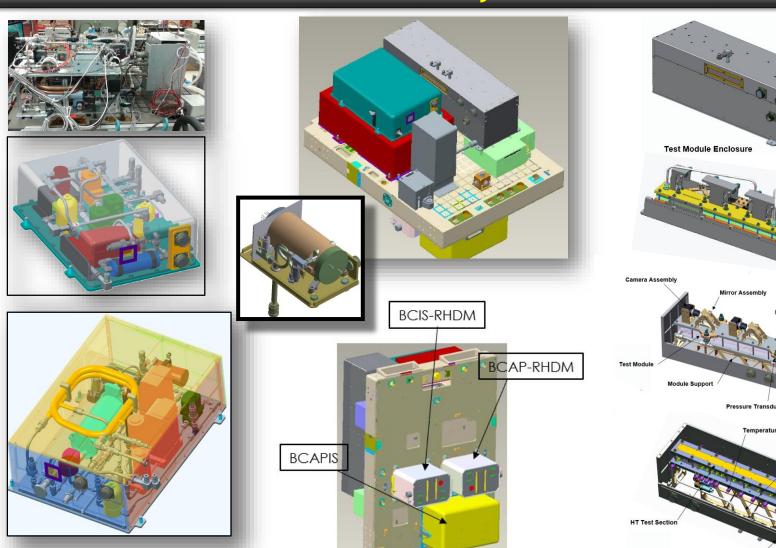


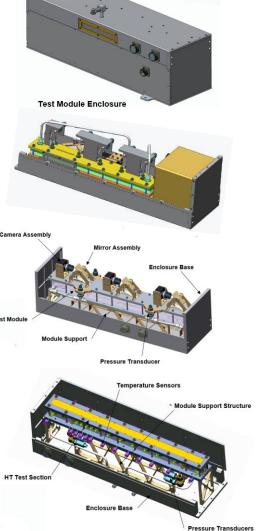
Flow condensation with increasing FC-72 flow rate and constant water flow rate



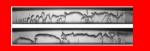


Summary









Concluding Remarks

- Flow Boiling and Condensation engineering team preparing for the Critical Design Review
- Design of flight hardware for FBCE
- Design of Engineering Models (EM) for CM-HT and Fluid system modules
- Development of Brassboard/EM hardware for future engineering testing of fluid system, software and avionics

Acknowledgement

 Authors acknowledge NASA's support to the Flow Boiling and Condensation Experiment

Questions??