

#### FACE SHEET/CORE DISBOND GROWTH IN HONEYCOMB SANDWICH PANELS SUBJECTED TO GROUND-AIR-GROUND PRESSURIZATION AND IN-PLANE LOADING

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The Aerospace Corporation, El Segundo, CA, June 25, 2015

Work was funded by NASA Langley Research Center under contract number NNL09AA00A

### **OVERVIEW**

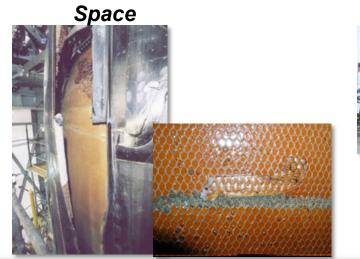


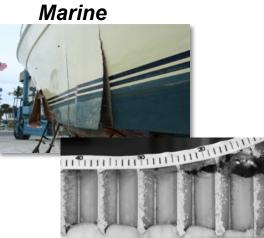
- Background
- Objective
- Detailed Problem Description
- Analysis Methodology
  - Fracture Mechanics Approach
  - Finite Element Modelling
- Initial Model Verification and Validation
- Analysis Results
  - Analysis of a Flat Panel Under Internal Pressure, In-Plane and Combined Loading
  - Analysis of a Curved Panel
- Summary
- Concluding Remarks

### BACKGROUND



- Problem
  - In-service component failures associated with disbonding in unvented honeycomb core sandwich
  - Degradation due to disbonding affects operational safety
  - Failures may discourage use of composites in 'future' vehicles
  - Methods for assessing propensity of sandwich structures to disbonding not fully matured, accepted and documented
  - Methods development is currently being discussed within the Disbond/ Delamination Task Group in CMH-17









### OBJECTIVE



- Identify, describe and address the phenomenon associated with face sheet/core disbonding
- Increase the knowledge on the subject and the awareness of consequences
- Develop a methodology to assess face sheet/core disbonding in honeycomb sandwich components similar to delamination in composite laminates
  - Develop standard test methods for characterizing face sheet/core disbonding in sandwich components
  - Develop a fracture mechanics based methodology to assess face sheet/ core disbonding in sandwich components
  - Develop models and analysis tools for face sheet/core disbonding in sandwich components subjected to ground-air-ground cycles and/or inplane loading
  - Evaluate the developed test methods and analysis tools using honeycomb sandwich panel tests

## DETAILED PROBLEM DESCRIPTION

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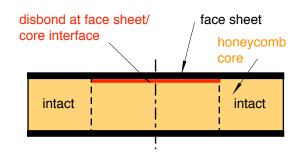


- Pressure difference between the in- and outside of unvented sandwich structures
  - Caused by alternating ambient pressure and temperature
  - Results in significant deformations and core volume increase
  - Volume increase results in pressure decrease based on the ideal gas law

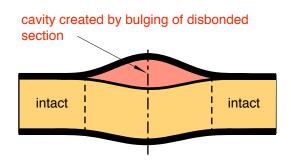
pV = nRT

- Initial disbonds between face sheets and core
  - increase the peeling effect and
  - decrease the structural reliability significantly
- For an accurate structural analysis, a coupled pressure-deformation problem needs to be solved

 Initial configuration at ground elevation



 Deformed configuration at cruising altitude



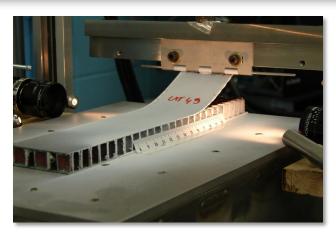
#### ANALYSIS METHODOLOGY Fracture Mechanics Approach

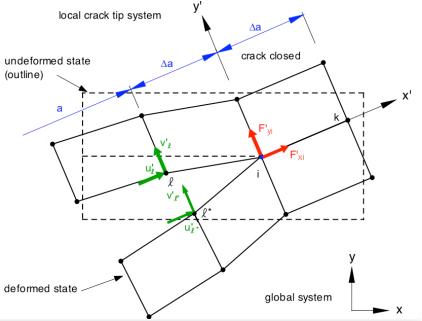
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- Two steps are required to identify, describe and address face sheet/ core disbonding
  - Test standard development in ASTM committee D30 (WK 47682)
    - Characterize the properties of the face sheet/core interface<sup>[14]</sup>
    - $\circ$  Measure fracture toughness G<sub>c</sub>
  - Analysis Development
    - Compute the energy release rate along the disbond front
    - Use the Virtual Crack Closure Technique (VCCT) based on the results obtained from a finite element analysis
- Propagation is predicted to occur once the computed value exceeds the measured fracture toughness

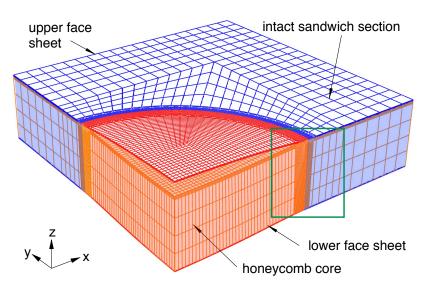
[14] reference to publication cited in conference proceedings

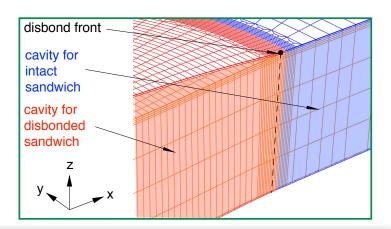




### ANALYSIS METHODOLOGY Finite Element Modelling – 1/4

- A quarter section of a flat panel was modelled
  - Circular disbond radius: 152.4 mm (6")
  - Square section modelled: 304.8 mm (12")
  - Abaqus/Standard<sup>®</sup> was used (C3D20)
    - Boundary conditions applied at symmetry planes
    - Surface contact used between top face sheet and core in the disbonded section
- Sandwich properties based on previous results
  - Thin face sheet: 0.772 mm (0.03")
    - CYCOM 5320PW plain weave fabric
    - [45/0/90/-45] quasi-isotropic layup
  - Thick core: 76.5 mm (3.0")
    - Hexcel HRH-10® honeycomb
    - NOMEX<sup>®</sup> paper with 48 kg/m<sup>3</sup> (3.0 lb/ft<sup>3</sup>) density and 3.175 mm (1/8") cell size
    - Modelled as an orthotropic, homogeneous continuum

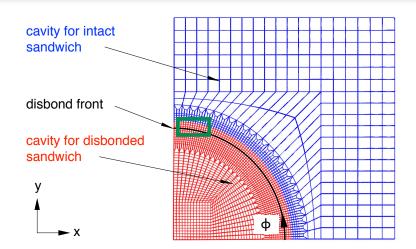


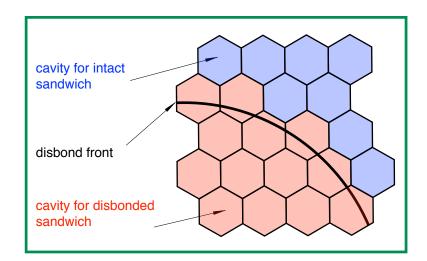


### ANALYSIS METHODOLOGY Finite Element Modelling – 2/4



- Pressure deformation coupling was simulated using fluid filled cavities
  - Abaqus/Standard<sup>®</sup> feature enabled the definition of fluid-filled cavities enclosed by structural elements
  - The ideal gas law is solved within each increment until equilibrium is found
  - The volume of the fluid cavities was assumed to be equal to that of the entire sandwich core
  - Two separate cavities were defined
    - One cavity was used to simulate the intact part
    - The other cavity included only the disbonded section
    - The disbonded cavity extended by one cell size, 3.175 mm (1/8"), ahead of the disbond front

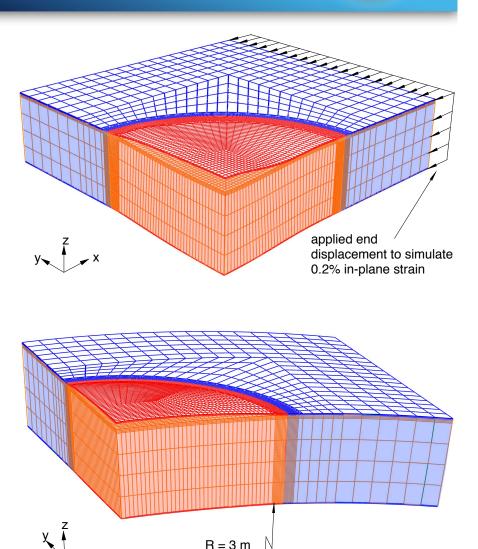




### ANALYSIS METHODOLOGY Finite Element Modelling – 3/4



- Model of a flat panel with in-plane loading
  - Study the effect of in-plane service load on a flat control surface
  - In-plane displacement applied to the model to simulate a 0.2% (2000 με) strain condition during a flight maneuver
  - A compressive strain condition was chosen since it was believed that it would aggravate the condition
- Model of a curved panel
  - Honeycomb sandwich constructions may be used for cylindrical fuselage structures
  - A 3 m radius (wide body airliner) was chosen for this study

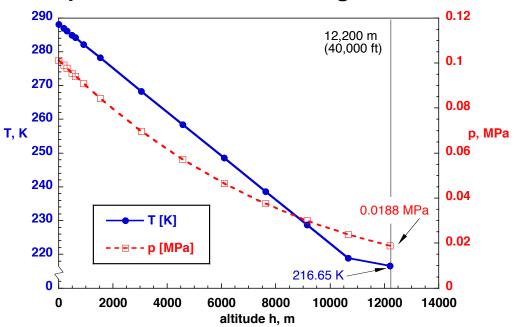


### ANALYSIS METHODOLOGY Finite Element Modelling – 4/4

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- Internal pressurization of the disbond
  - Commercial jetliner ascent scenario was considered from 0 to 12192 m (0 to 40000 ft).
  - The pressure and temperature values were taken from the International Standard Atmosphere ISO 2533
  - The temperature in the core was defined to be equal to the ambient temperature
  - Pressure and volume inside the cavities were calculated during the analysis
- Additional load conditions
  - 0.2% (2000  $\mu\epsilon$ ) strain condition only
  - Combination of GAG and 0.2%
    (2000 με) strain

Decrease of temperature and pressure with increasing altitude



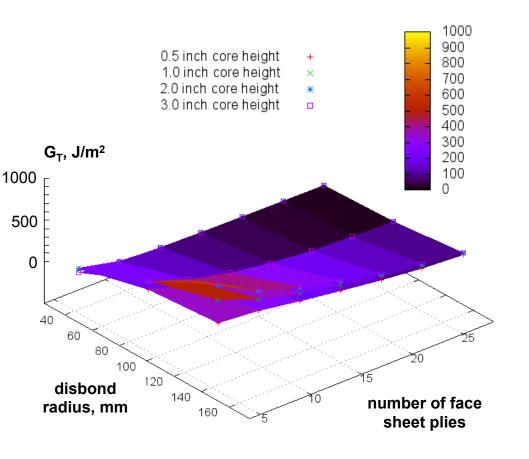
Flat panel under internal pressure loading – 1/3

- Initial study<sup>[6]</sup>
  - Variation of
    - Face sheet thickness, number of plies
    - o Disbond radius
    - Core density: 29 kg/m<sup>3</sup>, 48 kg/m<sup>3</sup>, 80 kg/m<sup>3</sup> (1.8 lb/ft<sup>3</sup>, 3.0 lb/ft<sup>3</sup>, 5.0 lb/ft<sup>3</sup>)
    - Core thickness: 12.5 mm,
      25.4 mm, 50.8 mm, 76.5 mm
      (0.5" 3.0")
  - Results
    - $\circ~$  Variation of core density does not have a significant effect on computed  $G_{\rm T}$
    - $\circ~$  Large disbond radius and thin face sheets result in maximum  $G_{T}$

#### Current study

 Dimensions based on results from initial study Averaged G<sub>T</sub> along crack front

3.275 mm (1/8") cell size, 48 kg/m<sup>3</sup> (3.0 lb/ft<sup>3</sup>) core density



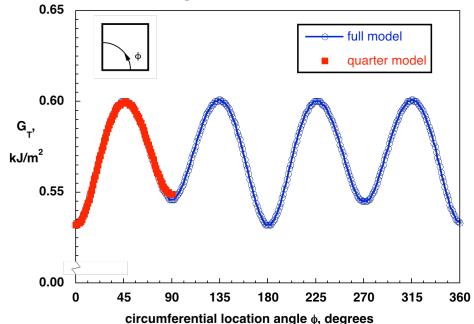


Flat panel under internal pressure loading – 2/3

#### Conditions

- 12,192 m altitude (40,000 ft)
  - External pressure p=0.0188 MPa
  - External temperature T= 216.65 K
- Verification for using a FE model of a quarter section of the panel
  - Analysis using a full model of the panel with circular disbond
  - Analysis using a model of a quarter panel with boundary conditions
  - Excellent agreement of computed  $G_T$  along the front for the currently used quasi-isotropic layup
  - Deviation, however, for other layups that violate the symmetry conditions of the model

• Distribution of energy release rate along the disbond front



Flat panel under internal pressure loading – 3/3

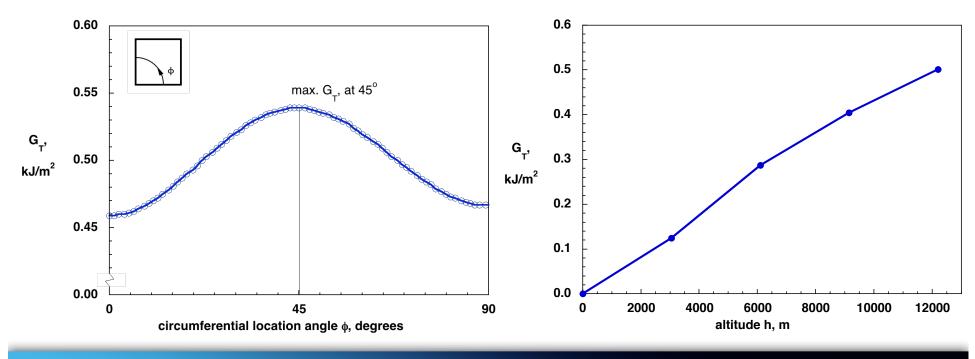
- **Conditions** •
  - 12,192 m altitude (40,000 ft)
    - External pressure p=0.0188 MPa
    - External temperature T= 216.65 K 0
- Result •
  - Max G<sub>T</sub> observed at φ=45°

#### **Conditions** •

- 0 m 12,192 m altitude
- Sea level to cruising altitude

#### Results for max $G_{\tau}$ at $\phi$ =45° •

 $G_{T}$  increases monotonically with increasing altitude



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Flat panel under in-plane and combined loading

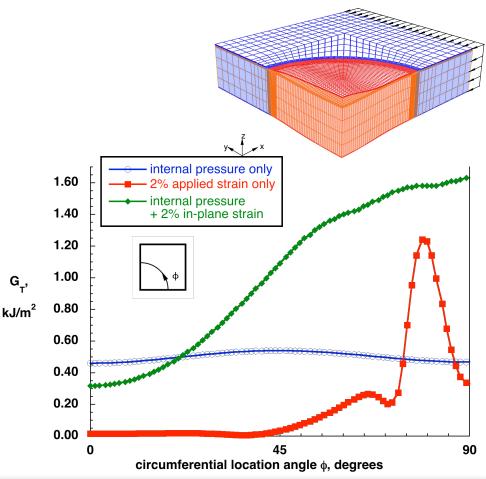
#### Conditions

- 12,192 m altitude (40,000 ft)
  - External pressure p=0.0188 MPa
  - External temperature T= 216.65 K
- 0.2% (2000 με) applied in-plane strain to simulate service loads on a flat control surface
- Combined internal pressure + 0.2%
  (2000 με) in-plane strain

#### Results

- Out of plane deformation of the disbonded section changes
- Leads to a change in the G<sub>T</sub> distribution
- In-plane strain aggravates the condition
- Due to non-linearity superposition of the results is not possible

• Distribution of energy release rate along the disbond front



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#### Analysis of a curved panel

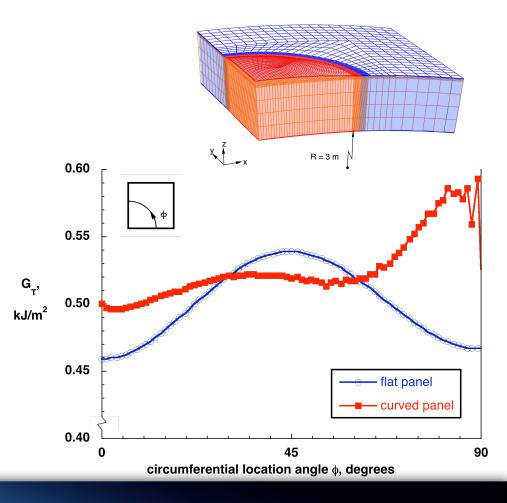
#### Conditions

- 12,192 m altitude (40,000 ft)
  - External pressure p=0.0188 MPa
  - External temperature T= 216.65 K
- Flat panel
- Curved panel with 3 m radius

#### Results

- Symmetry of the G<sub>T</sub> distribution is lost for the curved panel
- Locally and on average the computed G<sub>T</sub> is higher than the result obtained from the flat panel
- Result is unexpected
- In-plane strain may further aggravate the condition
- Additional analyses with different radii and more refined mesh should be preformed before a definite statement is made

• Distribution of energy release rate along the disbond front



### SUMMARY



- A sandwich panel containing a circular disbond at the face sheet/core interface was studied.
- A fracture mechanics approach was used.
- The pressure-deformation coupling was a focus of the analysis.
- Special fluid-filled cavities were used to model the entrapped air.
- Sandwich panels with large disbonds, thin face sheets, and thick cores are most critical.
- Computed averaged energy release rate values increased almost linearly with increasing altitude.
- The presence of the in-plane compressive strain aggravated the condition along the crack front.
- Due to the non-linearity of the problem, the results for combined load cases cannot simply be obtained by superposition of the individual load cases.
- For a curved panel with 3 m radius, the computed energy release rate values were higher than the values computed for a flat panel.

### **CONCLUDING REMARKS**



- Overall, the finite element analysis with fluid cavities appears to perform well and is capable of capturing the pressure-deformation coupling in the disbonded section of the panel.
- Based on the current preliminary results, however, it is recommended that additional validation studies be performed to compare.
  - The computed local deformation field of the disbonded face sheet with far field measurements
  - The computed pressure inside the cavity with measured values.
- Additionally, analyses of curved panels with different radii should be performed before a definite statement about the effect of panel curvature on the crack tip loading is made.
- Methods development will continue within the Disbond/Delamination Task Group in CMH-17

### ACKNOWLEDGEMENTS

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The analyses were performed at the Durability, Damage Tolerance and Reliability Branch at NASA Langley Research Center, Hampton, Virginia, USA while Zhi Chen was a participant in the Langley Aerospace Research Student Scholars (LARSS) program. Ronald Krueger (NIA) was supported under contract NNL09AA00A and Martin Rinker was a visiting scientist at the National Institute of Aerospace (NIA).



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### **BACKUP SLIDES**



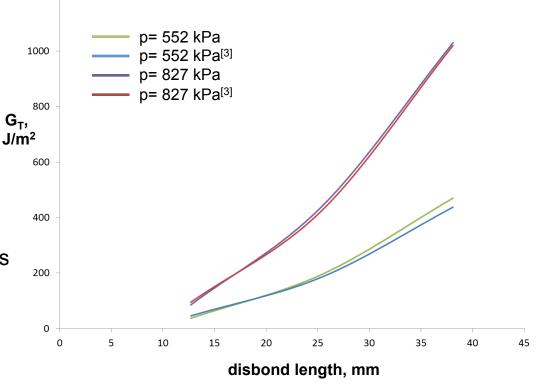
### INITIAL MODEL VERIFICATION AND VALIDATION – 1/2

1200



- X-33 cryogenic fuel tank
  - NASA sandwich disbond investigation<sup>[3]</sup>
    - Square delamination
    - Panel pressurized by a compressor
    - Defined load, no pressuredeformation coupling
    - Calculations were performed using surface loads
  - Current analysis approach<sup>[6]</sup>
    - Same dimensions as NASA publication
    - Pressure application with Abaqus fluid elements
    - VCCT calculation using postprocessing routine

- Result comparison
  - $\circ \quad \mbox{Good correlation between } G_{\rm T} \mbox{ values } \\ \mbox{ calculated using different models } \\$



## INITIAL MODEL VERIFICATION AND VALIDATION – 2/2

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#### Sandwich panel with disbond

- Airbus test in vacuum chamber<sup>[4]</sup>
  - Panel with 350 mm disbond
  - Pressure-deformation coupling needs to be considered
  - Pressure in disbonded core section was measured during test
  - FE analysis was performed calculating pressure-deformation coupling iteratively



- Current analysis approach
  - Same dimensions as Airbus panel
  - Pressure pressure-deformation
    coupling solved with Abaqus fluid
    elements
- Result comparison
  - Pressure-deformation coupling is correctly solved via Abaqus Fluid Cavity Simulation
  - Pressure in core:
    - Airbus test: 0.0582 MPa
    - $\circ$  Airbus analysis: 0.0577 MPa
    - o Current analysis: 0.0571 Mpa
- Additional validation studies should be performed to compare test results and analysis
  - Compare deformation field
  - Compare pressure inside the cavity