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# EXERGETIC ANALYSIS OF AN AIRCRAFT TURBOJET ENGINE WITH AN AFTERBURNER

by

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An exergy analysis is reported of a J85-GE-21 turbojet engine and its components for two altitudes: sea level and 11,000 meters. The turbojet engine with afterburning operates on the Brayton cycle and includes six main parts: diffuser, compressor, combustion chamber, turbine, afterburner, and nozzle. Aircraft data are utilized in the analysis with simulation data. The highest component exergy efficiency at sea level is observed to be for the compressor, at 96.7%, followed by the nozzle and turbine with exergy efficiencies of 93.7 and 92.3%, respectively. At both considered heights, reducing of engine intake air speed leads to a reduction in the exergy efficiencies of all engine components and overall engine. The exergy efficiency of the turbojet engine is found to decrease by 0.45% for every 1 °C increase in inlet air temperature.

Key words: turbojet, afterburner, exergy, entropy generation, efficiency

#### Introduction

Engines are installed on aircraft for propulsion. Exhaust gases from aircraft engines exit at the rear of the engine nozzle, causing the aircraft to move forward and air to pass over the aircraft wings. This air motion creates lift. Most modern aircraft use gas turbine engines to produce the required thrust force. Such engines are relatively light and compact and have a high power to weight ratios. Aircraft gas turbines operate in an open cycle based on the Brayton cycle. Actual jet cycles are not ideal due to thermodynamic losses. The gases in the jet cycle are expanded in the gas turbine to the pressure that allows generation of the power needed by the compressor, generator, hydraulic pump, and other work-consuming devices. Other gas turbine-based aerospace engines exist beyond the turbojet, including turbofan and turboshaft engines, but only turbojets are considered here. Production of jet aircraft began in 1945, and the technology has advanced significantly since then. Efforts to increase power and reduce noise and fuel consumption have led to improvements in turbojet and turbofan engines [1]. Nearly 16,800 jet aircraft were operating globally in 2007, and this number is expected to increase to 35,300 by 2024 [2]. Due to decreasing world oil reserves, increasing oil prices and increasing environmental concerns, efforts have increased to improve system efficiency. Exergy analysis is a practical and useful tool for such activities, with many engineers and scientists suggesting that exergy analysis is a highly effective method for evaluating and enhancing thermodynamic performance, and superior to energy analysis [3].

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Many researchers have investigated aircraft engines and propulsion systems with exergy analysis [4-12]. Rosen and Etele [13, 14] examine the importance of defining a reference environment that varies with altitude and apply that work to a turbojet engine. Gaggioli and Paulus [15] propose a method to account for the exergy rate associated with thrust and lift forces and develop aircraft exergy flow diagrams. Bejan and Siems [16] address the optimization of the cruise velocity and of the heat exchange processes on board an aircraft while Rancruel and Von Spakovsky [17] investigate engine configuration optimization methods. Hunt et al. [18, 19], and Riggins et al. [20] discuss methodologies for accounting for vehicle wake effects, and Moorhouse and Hoke [21] apply exergy methods to the analysis of the inlet performance. More recently, exergy analysis has been applied to an advanced hypersonic vehicle, with two main scopes: (1) to enhance the exergy approach to the design and optimization of aerospace propulsion systems, through the use of exergy flow diagram that provides aircraft engineers and system designers with insights into avoidable and unavoidable systemic losses, and (2) to explore the limits and merits of fuelling options for a scramjet-powered aircraft [22]. Turgut et al. [23] perform an exergoeconomic analysis of an aircraft turbofan engine utilising kerosene as fuel. They develop several parameters (thrust cost rate, cost of exergy destruction, relative cost difference, and exergoeconomic cost) and calculate the thrust cost rate to be 304.35 (\$/h kN) and 138.96 (\$/h kN) for hot and cold thrust, respectively. Turgut et al. [24] apply exergy analysis to a General Electric CF6-80 turbofan engine using sea-level data, and find the units with the greatest irreversibilities in the system to be the fan and the core engine exhaust, which exhibit exergy loss rates of 47.3 MW and 35.9 MW, respectively, and the combustion chamber, which has an exergy destruction rate of 31.5 MW. Previous studies have not examined turbojets with afterburning and have not focused on the variations of the results with altitude. Such information is needed and addressing these needs is the focus of this paper. The main objective is to perform an exergy analysis of a J85-GE-21 turbojet engine and its components, for two altitudes: sea level and 11,000 meters. Entropy generation and exergy efficiency relations are derived for each turbojet engine component and the overall system. The effects are studied of jet velocity and inter air temperature, on entropy generation and exergy efficiency, with the aid of a simulation program developed for this study.

## System modeling and analysis

The J85-GE-21 turbojet engine is designed for F-5 fighter aircraft models F and E by General Electric in the US and now is used on this aircraft. A J85-GE-21 turbojet engine is shown in fig. 1, where the main parts are: diffuser, compressor, combustion chamber, turbine, afterburner, and nozzle.

The turbojet has an axial compressor with nine stages, each having a 1.2 compression ratio. The average compression ratio of compressor is 9. The combustion chamber is of an annular type, and has 12 fuel injectors which spray liquid fuel (JET A-1) into the combustion chamber at high pressure. The turbine has two stages and produces the mechanical power needed by the compressor, hydraulic pump, and generator. Twenty fuel injectors spray liquid fuel into the afterburner, where it reacts with all the oxygen in the turbine exhaust gas in to order produce more power and propulsive force. Approximately 15% of the air exiting the diffuser passes around engine for cooling purposes, and is called dead air. The remaining air enters the compressor (40% of the air exiting the compressor goes around combustion chamber and afterburner to keep the flame in the center of these devices). Exhaust gas from turbojet engine is expanded to medium pressure by the nozzle Northrop [25]. Figure 2 shows a schematic diagram of the turbojet engine analyzed.

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Figure 1. Turbojet engine; used by permission of NASA (see website: www.Nasa.gov)



Figure 2. Schematic diagram of turbojet engine

The entry conditions of the diffuser  $(T_1, P_1, V_1)$  are known, and the following equations are used to obtain the outlet conditions. Note that outlet diffuser temperature is measured by a sensor installed at the end of diffuser to determine amount of fuel entering the combustion chamber and burning by the engine. The relevant basic equations are as follows:

$$h_0 = h + \frac{V^2}{2}$$
 (1),  $h = u + Pv$  (2),  $\eta_D = \frac{h_{02s} - h_1}{h_{01} - h_1}$  (3)

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where  $h_0$  [Jkg<sup>-1</sup>] is the denotes inlet diffuser specific stagnation enthalpy, h [Jkg<sup>-1</sup>] – the specific enthalpy, V [ms<sup>-1</sup>] – the velocity, u [Jkg<sup>-1</sup>] – the specific internal energy, P [Pa] – the pressure, v [m<sup>3</sup>kg<sup>-1</sup>] – the specific volume,  $\eta_D$  [Jkg<sup>-1</sup>] – the diffuser efficiency,  $h_{0s}$  [Jkg<sup>-1</sup>] – the outlet specific stagnation enthalpy for an isentropic process, and  $h_1$  [Jkg<sup>-1</sup>] – the inlet diffuser specific enthalpy. With the above relations, following equations can be written for the diffuser:

$$h_{01} = h_{02}$$
, (4),  $V_2 = 0$  (5),  $u_1 + P_1 v_1 + \frac{V_1^2}{2} = u_2 + P_2 v_2 + \frac{V_2^2}{2}$  (6)

where  $h_{01}$ , and  $h_{02}$  [Jkg<sup>-1</sup>] are the diffuser inlet and outlet specific stagnation enthalpies,  $u_1$  and  $u_2$  [Jkg<sup>-1</sup>] – the diffuser inlet and outlet specific internal energies,  $P_1$  and  $P_2$  [Pa] – the diffuser inlet and outlet pressures,  $V_1$  [ms<sup>-1</sup>] – the aircraft velocity,  $V_2$  [ms<sup>-1</sup>] – the outlet diffuser air speed, and  $v_1$  and  $v_2$  [m<sup>3</sup>kg<sup>-1</sup>] – the diffuser inlet and outlet specific volumes . With these equations, the diffuser outlet pressure and temperature can be obtained iteratively. The outlet compressor pressure is obtained with the following equation:

$$P_4 = r_c P_3 \tag{7}$$

where  $P_4$  and  $P_3$  [Pa] are the compressor outlet and inlet pressures, and  $r_c$  is the compressor pressure ratio. Also, the outlet compressor temperature is obtained with the following expressions:

$$dp = Tds + vdP (8), \quad dh_s = vdP (9), \quad \eta_C = \frac{dh_s}{dh} (10)$$

Here, dh,  $[Jkg^{-1}] ds [Jkg^{-1}K^{-1}]$ , and dP [Pa] are the specific enthalpy, specific entropy and pressure variations in the compression process, respectively, T[K] – the average temperature in the compression process, dh<sub>s</sub>  $[Jkg^{-1}]$  – the specific enthalpy variation for isentropic compression, and  $\eta_{\rm C}$  – the compressor polytropic efficiency. With the ideal gas assumption and eqs. 8 through 10, the following equation can be developed for the outlet temperature:

$$\overline{C}_{\rm P} \, \frac{\mathrm{d}T}{T} = \frac{R}{\eta_{\rm C}} \frac{\mathrm{d}P}{P} \tag{11}$$

Here,  $\overline{C}_{\rm P}$  [Jkg<sup>-1</sup>K<sup>-1</sup>] – the specific heat at constant pressure, which is evaluated at a mean temperature, and *R* [Jkg<sup>-1</sup>K<sup>-1</sup>] – the gas constant. With the specific heat at constant pressure as a function of temperature and the compressor pressure ratio, the compressor outlet temperature can be obtained for gas mixtures by integration of above equation. The integral form of above equation is:

$$\int_{T_3}^{T_4} \overline{C}_P \frac{\mathrm{d}T}{T} = \int_{P_3}^{P_4} \frac{R}{\eta_C} \frac{\mathrm{d}P}{P} = \frac{R}{\eta_C} \int_{P_3}^{P_4} \frac{\mathrm{d}P}{P}$$
(12)

Here,  $T_3$  and  $T_4$  [K] are inlet and outlet compressor temperature and  $P_3$  and  $P_4$  [Pa] are inlet and outlet compressor pressure. The compressor power input  $W_C$  is calculated as:

$$W_C = \dot{m}_a \left( h_4 - h_3 \right) - 0.4 \dot{m}_a h_{12} \tag{13}$$

Here,  $\dot{m}_a$  [kgs<sup>-1</sup>] is air mass flow rate and  $h_3$  and  $h_4$  and  $h_{12}$  [Jkg<sup>-1</sup>] are inlet and outlet specific enthalpies of the compressor. Kerosene is a fuel currently used in aviation. This fuel can have variable functional applications by modification of some of its properties. Two types of kerosene are Jet A and Jet A-1 [26, 27]. Depending on the type of ball bearings, roll bearings, sealing, and packing of selected engines, Jet A-1 or JP4 is normally used [25]. According to fig. 2, there are two components in which combustion occurs. Combustion relations based on reactant and product mass balances are [28]: - combustion chamber:

$$0.0024C_{12}H_{23} + 0.4944(0.7748N_2 + 0.2059O_2 + 0.0003CO_2 + 0.019H_2O) \rightarrow 0.02889CO_2 + 0.03694H_2O + 0.05928O_2 + 0.38306N_2$$
(14)

- afterburner:

$$0.0073C_{12}H_{23} + 1.189521(0.762236N_2 + 0.152502O_2 + 0.034138CO_2 + 0.151124H_2O) \rightarrow 0.1228CO_2 + 0.26371H_2O + 0.072562O_2 + 0.9064N_2$$
(15)

The energy balance for a combustion process can be written as:

$$\sum \dot{m}_{\rm r} \left[ \dot{h}_{\rm f} + (h - h_0) \right]_{\rm p} = \eta_{\rm cc} \sum \dot{m}_{\rm p} \left[ \dot{h}_{\rm f} + (h - h_0) \right]_{\rm r}$$
(16)

where  $h_f$  [Jkg<sup>-1</sup>] is the specific enthalpy of formation,  $h_0$  [Jkg<sup>-1</sup>] – the specific enthalpy at the reference condition,  $\eta_{cc}$  – the combustion energy efficiency, and the subscripts r and p denote reactants and products, respectively. The combustion temperature can be calculated with eqs. (14) to (16). The pressure in the combustion chamber can be determined as:

$$P_6 = P_5 \frac{n_6}{n_5} \frac{T_6}{T_5} \tag{17}$$

where  $P_5$  and  $P_6$  [Pa] are the inlet and outlet pressures of the combustion chamber,  $T_5$  and  $T_6$  [K] – the inlet and outlet temperatures, and  $n_6$  and  $n_5$  are the inlet and outlet number of moles. Similarly, the pressure in the afterburner can be expressed as:

$$P_8 = P_7 \frac{n_8}{n_7} \frac{T_8}{T_7} \tag{18}$$

where  $P_7$  and  $P_8$  are the inlet and outlet pressures of the afterburner,  $T_7$  and  $T_8$  are the inlet and outlet temperatures, and  $n_7$  and  $n_8$  are the inlet and outlet number of moles. The turbine outlet pressure can be expressed as:

$$P_7 = r_{\rm T} P_6 \tag{19}$$

where  $P_7$  and  $P_6$  [Pa] are the turbine outlet and inlet pressures, and  $r_T$  – the turbine pressure ratio. The turbine outlet temperature can be obtained by the equation:

$$\int_{T_6}^{T_7} \overline{C}_P \frac{\mathrm{d}T}{T} = \int_{P_6}^{P_7} \frac{R}{\eta_{\mathrm{T}}} \frac{\mathrm{d}P}{P} = \frac{R}{\eta_{\mathrm{T}}} \int_{P_6}^{P_7} \frac{\mathrm{d}P}{P}$$
(20)

here,  $T_6$  and  $T_7$  are the inlet and outlet turbine temperatures,  $P_6$  and  $P_7$  are inlet and outlet turbine pressures, and  $\eta_T$  – the turbine polytropic efficiency. The turbine power production  $\dot{W}_T$  can be expressed as:

$$W_{\rm T} = (\dot{m}_{\rm a} + \dot{m}_{\rm f})(h_6 - h_7)$$
 (21)

where  $\dot{m}_a$  is the air mass flow rate [kgs<sup>-1</sup>],  $\dot{m}_f$  [kgs<sup>-1</sup>] – the fuel mass flow rate, and  $h_6$  and  $h_7$  [Jkg<sup>-1</sup>] are the inlet and outlet specific enthalpies of the turbine. The nozzle outlet temperature can be determined using the equations:

$$h_{08} = h_{09}$$
 (22),  $V_8 = 0$  (23),  $u_8 + P_8 v_8 \frac{V_8^2}{2} = u_9 P_9 v_9 + \frac{V_9^2}{2}$  (24)

Here,  $h_{08}$ , and  $h_{09}$  [Jkg<sup>-1</sup>] are the inlet and outlet nozzle specific stagnation enthalpies,  $u_8$ , and  $u_9$  [Jkg<sup>-1</sup>] – the inlet and outlet nozzle specific internal energies,  $P_8$  and  $P_9$  [Pa] – the inlet and outlet nozzle pressures,  $V_8$  [ms<sup>-1</sup>] – the inlet nozzle velocity,  $V_9$  [ms<sup>-1</sup>] – the outlet diffuser air speed, and  $v_8$  and  $v_9$  [m<sup>3</sup>kg<sup>-1</sup>] are the nozzle inlet and outlet specific volumes.

#### **Exergy analysis**

In the absence of nuclear, magnetic, electric, and surface tension effects, exergy can be divided into four distinct components: physical, chemical, kinetic, and potential [3, 28, 29]:

$$e_{\rm t} = e_{\rm kn} + e_{\rm pt} + e_{\rm ph} + e_{\rm ch} \tag{25}$$

where  $e_t$ ,  $e_{kn}$ ,  $e_{pt}$ ,  $e_{ph}$ , and  $e_{ch}$  [Jkg<sup>-1</sup>] are the total, kinetic, potential, physical, and chemical specific exergises. Specific kinetic and potential exergises can be expressed as:

$$e_{\rm kn} = \frac{V^2}{2}$$
 (26),  $e_{\rm pt} = gh$  (27)

where g  $[ms^{-2}]$  is the gravitational acceleration and h [m] – the height. Specific physical and chemical exergises can be written as:

$$e_{\rm ph} = (h - h_0) - T_0(s - s_0)$$
 (28),  $e_{\rm ch} = \sum_i x_i e_{ch,i} + RT_0 \sum x_i Lnx_i$  (29)

where h [Jkg<sup>-1</sup>] denotes the specific enthalpy,  $h_0$  [Jkg<sup>-1</sup>] – the specific enthalpy at the reference condition, s [Jkg<sup>-1</sup>] – the specific entropy,  $s_0$  [Jkg<sup>-1</sup>K<sup>-1</sup>] – the specific entropy at the reference condition,  $T_0$  [K] – the reference temperature,  $x_i$  – the gas mole fraction, and  $e_{chi}$  – specific chemical exergy. Exergy rates [W] can be determined with the equations:

$$\dot{E}_{\rm kn} = \dot{m}e_{\rm kn}$$
 (30),  $\dot{E}_{\rm pt} = \dot{m}e_{\rm pt}$  (31),  $\dot{E}_{\rm ph} = \dot{m}e_{\rm ph}$  (32),  $\dot{E}_{\rm ch} = \dot{m}e_{\rm ch}$  (33)

where  $E_{\rm kn}$ ,  $E_{\rm pt}$ ,  $E_{\rm ph}$ , and  $E_{\rm ch}$  are the kinetic, potential, physical, respectively, and chemical exergy rates  $\dot{m}$  and is the mass flow rate.

Since the diffuser is used to increase pressure and reduce velocity and involves no power interaction, the diffuser exergy efficiency can be expressed as:

$$\Psi_{\text{Diffuser}} = \frac{E_2}{\dot{E}_1} \tag{34}$$

where  $\Psi$  denotes the exergy efficiency,  $\dot{E}_2$ , and  $\dot{E}_1$  [W] are outlet and inlet diffuser exergy rates. The compressor exergy efficiency is calculated as:

$$\Psi_{\rm Compressor} = \frac{\dot{E}_{12} + \dot{E}_4 - \dot{E}_3}{\dot{W}_{\rm C}}$$
(35)

where  $\dot{E}_4$ ,  $\dot{E}_3$ , and  $\dot{E}_{12}$  [W] are the outlet and inlet compressor exergy rates. The combustion chamber exergy efficiency is expressible as:

$$\Psi_{\text{Combustion chamber}} = \frac{\dot{E}_6}{\dot{E}_{13} + \dot{E}_5 + \dot{E}_{10}} \tag{36}$$

where  $\dot{E}_6$  [W] is the outlet combustion chamber exergy rate,  $\dot{E}_5$  [W] – the air inlet combustion chamber exergy rate,  $\dot{E}_{10}$  [W] – the fuel exergy consumption rate in the combustion chamber, and  $\dot{E}_{13}$ [W] – the air exergy consumption rate in the combustion chamber. The turbine exergy efficiency can be written as:

$$\Psi_{\text{Turbine}} = \frac{W_t}{\dot{E}_6 - \dot{E}_7} \tag{37}$$

where  $\dot{E}_7$  [W] is the outlet turbine exergy rate. The afterburner exergy efficiency can be expressed as:

$$\Psi_{\text{Afterburner}} = \frac{E_8}{\dot{E}_{14} + \dot{E}_{11} + \dot{E}_7}$$
(38)

where  $\dot{E}_8$  [W] is the outlet afterburner chamber exergy rate,  $\dot{E}_{11}$  [W] – the fuel use exergy rate in the afterburner, and  $\dot{E}_{14}$  [W] – the air use exergy rate in the afterburner. The nozzle exergy efficiency is determined as:  $\dot{E}$ 

$$\Psi_{\text{Nozzle}} = \frac{E_9}{\dot{E}_8} \tag{39}$$

where  $\dot{E}_9$  [W] is the outlet nozzle exergy rate. The overall turbojet exergy efficiency can be written as:  $\dot{E}$ 

$$\Psi = \frac{E_9}{\dot{E}_1 + \dot{E}_{10} + \dot{E}_{11}} \tag{40}$$

The entropy generation rates in the diffuser, compressor, combustion chamber, turbine, afterburner, and nozzle, can be expressed as:

$$\dot{S}_{\text{gen, diffuser}} = \frac{1}{T_o} (\dot{E}_2 - \dot{E}_1) \tag{41}$$

$$\dot{S}_{\text{gen, compressor}} = \frac{1}{T_{o}} [\dot{E}_{12} + \dot{E}_{4} - \dot{E}_{3} - (-\dot{W}_{c})]$$
(42)

$$\dot{S}_{\text{gen, compressor chamber}} = \frac{1}{T_{o}} (\dot{E}_{13} + \dot{E}_{10} + \dot{E}_{5} - \dot{E}_{6})$$
 (43)

$$\dot{S}_{\text{gen, turbine}} = \frac{1}{T_0} (\dot{E}_6 - \dot{E}_7 - \dot{W}_t)$$
 (44)

$$\dot{S}_{\text{gen, afterburner}} = \frac{1}{T_{o}} (\dot{E}_{7} + \dot{E}_{14} + \dot{E}_{11} - \dot{E}_{8})$$
 (45)

$$\dot{S}_{\text{gen, nozzle}} = \frac{1}{T_0} (\dot{E}_8 - \dot{E}_9)$$
 (46)

where  $T_{0}$  is the reference environment temperature, which is fixed here at 298.15 K.

## **Results and discussion**

Temperatures, pressures, mass flow rates, kinetic, physical and chemical exergy rates for various parts of engine at sea level and 200 m/s aircraft velocity are shown in tab. 1.

Exergy efficiencies and entropy generation rates for the components of the engine at sea level and 200 m/s speed are shown in figs. 3 and 4, respectively. The highest component exergy



140 140 120 120 100 80 Sgen 54.8 60 40 14.5 20 7.64 2.093 0.015 0.1 0 Turbine Diffuser Combustion Afterburner Nozzle Compressor

Figure 3. Exergy efficiencies for turbojet engine and its components at sea level and 200 m/s speed

Figure 4. Entropy generation rates for turbojet engine and its components at sea level and 200 m/s speed

Flow No.	Flow	Temperature [K]	Pressure [kPa]	Mass flow rate [kgs <sup>-1</sup> ]	Physical exergy rate [kW]	Chemical exergy rate [kW]	Kinetic exergy rate [kW]	Total exergy rate [kW]
1	Diffuser inlet	298.1	101.3	28.4	0	0	568.2	568.2
2	Diffuser outlet	318.1	124.2	28.4	514.6	0	0	514.6
3	Compressor inlet	318.1	124.2	24.1	437.4	0	0	437.4
4	Compressor outlet	625.7	1118.2	24.1	7679.5	0	0	7679.5
5	Combustion chamber inlet	625.7	1118.2	14.5	4607.7	0	0	4607.7
6	Turbine inlet	1709.6	2138.8	14.9	18409.2	0	0	18409.2
7	Afterburner inlet	1184	534.7	14.9	10297.4	0	0	10297.4
8	Afterburner outlet	2933.5	1263.8	19	36170.4	0	0	36170.4
9	Nozzle outlet	2866.1	101.3	19	32781.3	0	1111.1	33892.4
10	Combustion chamber fuel input	353	2757	0.4	0	18282.3	0	18282.3
11	Afterburner fuel input	353	1103.1	218.1	0	55688	0	55688
12	Compressor cooling output	625.7	1118.2	9.6	3071.8	0	0	3071.8
13	Combustion chamber cool- ing air input	625.7	1118.2	5.8	1843.1	0	0	1843.1
14	Afterburner cooling air input	625.7	1118.2	3.8	1228.7	0	0	1228.7

Table 1. Thermodynamic parameters for flows in J85GE-21 turbojet engine at sea level and 200 m/s speed

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efficiency is exhibited by the compressor at 96.7%. The next highest exergy efficiencies are for the nozzle (93.7%) and turbine (92.3%). The lowest exergy efficiencies are for the afterburner (54.8%), followed by the combustion chamber (80.4%). The low fuel efficiency of the afterburner compared to the combustion chamber results in it burning almost three times the fuel of the combustion chamber, in order to increase the thrust force by about one third. The entropy production rates also help identify the components with the highest irreversibilities. The greatest entropy production rate in the engine is observed for the afterburner, followed by the combustion chamber. Hence, the combustion processes in the aircraft engine are highly irreversible. The nozzle, due to the rapid changes in cross-sectional area has in the third highest entropy production rates.

To determine the effect of aircraft engine velocity on entropy production rate and exergy efficiency, the engine velocity is reduced from 200 m/s to 100 m/s and the results re-determined (see tab. 2). For this case, the exergy efficiencies and entropy generation rates are shown in figs.

Flow No.	Flow	Temperature [K]	Pressure [kPa]	Mass flow rate [kgs <sup>-1</sup> ]	Physical exergy rate [kW]	Chemical exergy rate [kW]	Kinetic exergy rate [kW]	Total exergy rate [kW]
1	Diffuser inlet	298.1	101.3	14.2	0	0	71	71
2	Diffuser outlet	303	106.7	14.2	64	0	0	64
3	Compressor inlet	303	160.7	12.1	54.4	0	0	54.4
4	Compressor outlet	596.3	960.5	12.1	3483.6	0	0	3483.6
5	Combustion chamber inlet	596.3	960.5	7.2	2090.1	0	0	2090.1
6	Turbine inlet	1684.6	1899.4	7.6	9500.8	0	0	9500.8
7	Afterburner inlet	1166.7	474.9	7.6	5082.9	0	0	5082.9
8	Afterburner outlet	29.48.6	1144.9	9.9	19098.6	0	0	19098.6
9	Nozzle outlet	2880.2	101.3	9.9	17408.6	0	592	18000.6
10	Combustion chamber fuel input	353	2757	0.4	0	18282.3	0	18282.3
11	Afterburner fuel input	353	1103.1	1.2	0	55687.9	0	55687.9
12	Compressor cooling output	596.3	960.5	4.8	2090	0	0	2090
13	Combustion chamber cooling air input	596.3	960.5	2.9	1254	0	0	1254
14	Afterburner cooling air input	596.3	960.5	1.92	836	0	0	836

Table 2. Thermodynamic parameters for flows in J85GE-21 turbojet engine at sea level and 100 m/s speed

5 and 6. The exergy efficiencies the overall engine of all it components decrease as the engine velocity decreases due to the reduction air mass flow rate.

To evaluate the variation of the performance of aircraft engine and its components with altitude, the performance parameters considered earlier are evaluated for an altitude of 11,000 m, which is a typical altitude at which a majority of aircraft flight time is spent. The temperature and pressure at high altitudes are much different than at sea level. For instance, these parameters



Figure 5. Exergy efficiencies for turbojet engine and its components at sea level and 100 m/s speed



Figure 6. Entropy generation rates for turbojet engine and its components at sea level and 100 m/s speed

were measured as 220 K and 4.23 kPa, respectively, by the meteorological office on February 17, 2010, at 10 a. m. at an altitude 11,000 m. The temperatures, pressures, mass flow rates, and

Flow No.	Flow	Temperature [K]	Pressure [kPa]	Mass flow rate [kgs <sup>-1</sup> ]	Physical exergy rate [kW]	Chemical exergy rate [kW]	Kinetic exergy rate [kW]	Total exergy rate [kW]
1	Diffuser inlet	220.15	23.4	8.9	0	0	177.8	177.8
2	Diffuser outlet	240	30.8	8.9	161.3	0	0	161.3
3	Compressor inlet	240	3.8	7.6	137.1	0	0	137.1
4	Compressor outlet	472.23	276.9	7.6	1830.1	0	0	1830.1
5	Combustion chamber inlet	472.23	276.9	4.5	1098	0	0	1098
6	Turbine inlet	1586.6	651.2	4.7	6135.3	0	0	6135.3
7	Afterburner inlet	1098.8	162.8	4.7	3356.8	0	0	3356.8
8	Afterburner outlet	2823.5	399.1	5.8	12423.2	0	0	12423.2
9	Nozzle outlet	2777.7	23.4	5.8	11465.9	0	256.7	11722.6
10	Combustion chamber fuel input	320	2757	0.13	0	5745.2	0	5745.2
11	Afterburner fuel input	320	1103.2	0.38	0	17500	0	17500
12	Compressor cooling output	472.23	276.9	3.04	1098	0	0	1098
13	Combustion chamber cooling air input	472.23	276.9	1.8	658.8	0	0	658.8
14	Afterburner cooling air input	472.23	276.9	1.22	439.2	0	0	439.2

Table 3. Thermodynamic parameters for flows in J85GE – 21 turbojet engine at 11,000 m altitude and 200 m/s speed

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Flow No.	Flow	Temperature [K]	Pressure [kPa]	Mass flow rate [kgs <sup>-1</sup> ]	Physical exergy rate [kW]	Chemical exergy rate [kW]	Kinetic exergy rate [kW]	Total exergy rate [kW]
1	Diffuser inlet	220.1	23.4	4.4	0	0	22.2	22.2
2	Diffuser outlet	225.1	25.1	4.4	20	0	0	20
3	Compressor inlet	225.1	25.1	3.8	17	0	0	17
4	Compressor outlet	442.9	226	3.8	806.3	0	0	806.3
5	Combustion chamber inlet	442.9	226	2.3	483.8	0	0	483.8
6	Turbine inlet	1562.6	558.1	2.4	3367.2	0	0	3367.2
7	Afterburner inlet	1082.2	139.5	2.4	1659.9	0	0	1659.9
8	Afterburner outlet	2828.85	347.9	3.1	6588.2	0	0	6588.2
9	Nozzle outlet	2782.6	23.4	3.1	6106.9	0	136.8	6243.7
10	Combustion chamber fuel input	320	2757	0.13	0	5745.2	0	5745.2
11	Afterburner fuel input	320	1103.1	0.38	0	17499.9	0	17499.9
12	Compressor cooling output	442.9	226	2.28	483.78	0	0	493.8
13	Combustion chamber cooling air input	442.9	226	1.4	290.3	0	0	296.3
14	Afterburner cooling air input	442.9	226	0.88	193.5	0	0	197.5

Table 4. Thermodynamic parameters for flows in J85GE-21 turbojet engine at 11,000 m altitude and 100 m/s speed

kinetic, physical, and chemical exergy rates for various components of the engine at altitude of 11,000 m and a 200 m/s aircraft velocity are shown in tab. 3. Exergy efficiencies and entropy generation rates the engine and its components at an altitude of 11,000 m and an engine speed of 200 m/s are shown in figs. 7 and 8. The compressor has the highest exergy efficiency at 95.7%, followed by the diffuser (94.8%) and nozzle (90.5%).

To evaluate the variation of the performance of aircraft engine and its components with aircraft velocity at high altitude, the entropy production rates and exergy efficiencies are evaluated when the engine velocity is reduced from 200 m/s to 100 m/s at an altitude of 11,000 m The results are given in tab. 4. For this case, the exergy efficiencies and entropy generation rates are shown in figs. 9 and 10.



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Figure 7. Exergy efficiencies for turbojet engine and its components at 11,000 m altitude and 200 m/s speed



Figure 9. Exergy efficiencies for turbojet engine and its components at 11,000 m and 100 m/s speed



Figure 11. Variation of aircraft exergy efficiency on aircraft engine

### Conclusions



Figure 8. Entropy generation rates for turbojet engine and its components at 11,000 m altitude and 200 m/s speed



Figure 10. Entropy generation rates for turbojet engine and its components at 11,000 m and 100 m/s speed

The effect of inlet air temperature on exergy efficiency for the aircraft engine is shown in fig. 11. The overall exergy efficiency of the aircraft engine is reduced 0.45% for each centigrade degree increase in inlet air temperature. The cause of this phenomenon is the decrease in air density due to the increase in temperature. This observation suggests that flying in early hours of the day or at night is economic due to the lower ambient temperature.

The exergy analysis of the selected aircraft engine with afterburner operating on kerosene fuel for various altitudes and air speeds reveals useful insights, as does the investigation of the effects of aircraft inlet air temperature on exergy efficiency. The relations are quantified between exergy efficiency and entropy generation for the components of the aircraft engine. The key results and conclusions are:

- The highest component exergy efficiency is observed at sea level for the compressor (96.7%), followed by the nozzle (93.7%), and turbine (92.3%). Similarly, the highest exergy efficiency at an altitude of 11,000 m is exhibited by the compressor (95.7%), followed by the nozzle (94.8%), and the diffuser (90.5%).
- The lowest component exergy efficiency at sea level is observed for the afterburner (54.8%), followed by the combustion chamber (80.4%). Correspondingly, the entropy production rates at sea level indicate that the most irreversible process is the afterburner, followed by the combustion chamber and nozzle.
- Reducing aircraft velocity at sea level reduces the exergy efficiency of the aircraft engine and its components.
- The aircraft engine exergy efficiency is reduced 0.45% per centigrade degree increase in inlet air temperature.

## Nomenclature

$\begin{array}{c} C_p \\ \mathrm{d}h \\ \mathrm{d}h_{\mathrm{s}} \\ \dot{E}_{\mathrm{ch}} \end{array}$	<ul> <li>specific heat at constant pressure, [Jkg<sup>-1</sup>]</li> <li>specific enthalpy variation in process, [Jkg<sup>-1</sup>]</li> <li>specific enthalpy variation in isentropic process, [Jkg<sup>-1</sup>]</li> <li>chemical exergy rate, [W]</li> </ul>	v <sub>.</sub> W <sub>.C</sub> W <sub>T</sub> Greek	<ul> <li>specific volume, [m<sup>3</sup>kg<sup>-1</sup>]</li> <li>compressor input power, [W]</li> <li>turbine output power, [W]</li> <li>symbols</li> </ul>
<u>E</u> kn	<ul> <li>kinetic exergy rate, [W]</li> </ul>	$\eta_{ m c}$	<ul> <li>compressor polytropic efficiency</li> </ul>
E <sub>ph</sub>	- physical exergy rate, [W]	$\eta_{ m D}$	<ul> <li>diffuser polytropic efficiency</li> </ul>
$E_{\rm pt}$	- potential exergy rate, [W]	$\eta_{ m T}$	<ul> <li>turbine polytropic efficiency</li> </ul>
$e_{\rm t}$	- specific exergy, [Jkg <sup>-1</sup> ]	Ψ	<ul> <li>second law efficiency</li> </ul>
g	- gravitational acceleration, [ms <sup>-</sup> ]	Subac	rinto
h	- neight of aircraft, [m]	SUDSCI	ipis
h 1.	- specific enthalpy, $[JKg]$	ch	- chemical
$n_0$	- specific enthalpy at reference condition, $[Jkg]$	kn	– kinetic
$n_{\rm f}$	- specific entitling of formation, [Jkg ]	р	- product
$h_{01}$	- Inter specific stagnation enthalpy, [Jkg ]	ph	– physical
$h_{02}$	- outlet specific stagnation enthalpy, [JKg ]	pt	– potential
$n_{0S}$	isontropic process [Ikg <sup>-1</sup> ]	r	– reactant
	mass flow rate [kgs <sup>-1</sup> ]	1	<ul> <li>diffuser inlet</li> </ul>
<i>m</i>	- mass now rate, [kgs]	2	<ul> <li>diffuser out let</li> </ul>
m <sub>a</sub>	- all mass now rate, [kgs ]	3	<ul> <li>compressor Inlet</li> </ul>
m <sub>f</sub>	- Idel mass now rate, [kgs ]	4	<ul> <li>compressor outlet</li> </ul>
n D	- Infinite of moles	5	<ul> <li>combustion chamber inlet</li> </ul>
Г D	- pressure, [ra]	6	<ul> <li>combustion chamber outlet</li> </ul>
К И	- gas constant, [JKg K ]	7	<ul> <li>afterburner inlet or turbine outlet</li> </ul>
r <sub>c</sub>	- compressor pressure ratio	8	<ul> <li>afterburner outlet or nozzle inlet</li> </ul>
	- turbile pressure ratio	9	<ul> <li>nozzle outlet</li> </ul>
1	- mean temperature in compression	10	<ul> <li>combustion chamber inlet</li> </ul>
Т	plocess, [K]	11	<ul> <li>afterburner inlet</li> </ul>
1 <sub>0</sub>	- reference environment temperature, $[K]$	12	<ul> <li>compressor cooling air outlet</li> </ul>
u V	- specific internal energy, [JKg ]	13	- combustion chamber cooling air inlet
V IZ	- velocity, [IIIS ]	14	<ul> <li>afterburner cooling air inlet</li> </ul>
$V_1$	- aircrait velocity, [ms]		

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