# Harmony Search Optimization and Damage Tolerance of Structural Systems 

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## R. Bryan Peiffer

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## Date

Thesis Advisor

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The opinions, findings, and conclusions expressed in this thesis are those of the author's and do not necessarily reflect the views of the individuals acknowledged above.

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#### Abstract

The aim of this thesis is to develop a MATLAB computer model for optimum design of steel structures via harmony search (HS) algorithm. The objective of the optimization routine is to provide a minimum weight structure while satisfying prescribed constraints such as strength and displacement limitations. The HS algorithm is a relatively new optimization method that has shown promise when adapted to structural optimization problems. Unlike other optimization routines, limited research has been presented incorporating this mathematical model. The author of this thesis decided to test the applications of the HS algorithm in structural engineering problems.

The HS algorithm is a meta-heuristic search method recently developed and adapted to optimization problems. It uses a stochastic derivative, which utilizes the experiences of musicians in Jazz improvisation to find optimal solutions. It differs from classical calculus bases optimization techniques that require gradient information by giving each decision variable a probability of selection.

Three examples have been provided showing the capabilities of the HS for least weight optimization of truss and frame structures. Both continuous and discrete optimization routines are present in this thesis. In the discrete optimization routine, standard steel shapes were used in accordance to the American Institute of Steel Construction (AISC) shape database. Strength and displacement constraints from the 2005 AISC load and resistance factor design specification were used to design.


In addition to least weight optimization, damage tolerance optimization of structural systems was also considered. Least weight optimization was performed accounting for probable future damage to the structure. A general mathematical model for damage tolerant optimization is presented. This method is based on serviceability, ultimate and residual capacity requirements.

It is shown that the HS algorithm can be a powerful tool for optimization problems, particularly structural engineering optimization problems. It has the ability to handle complex problems that would be very challenging to solve by traditional methods. It also, has been shown to be competitive with several other well know meta-heuristic optimization methods.

## Chapter 1: Introduction

### 1.1 General

One of the most catastrophic structural failures in modern history has been the collapse of the World Trade Center towers in 2001. Most individuals do not think twice about the structural systems they encounter on a daily basis. They typically view structural assemblages such as the twin towers as invulnerable structures incapable of collapse. However, the collapse of the twin towers, among other failures, shed light to the public that structures are vulnerable.

The collapse of the twin towers was found to be from a pancaking action that resulted in the towers crushing themselves completely after the planes struck. The term progressive collapse was a widely used term in the structural engineering community after the events. While progressive collapse is not a new phenomenon, it has been a source of increased interest due to these large-scale failures. Because of these failures, increased attention has been focused upon the concepts of structural robustness, reliability, damage and redundancy. The importance of design procedures that provide redundant and robust structures is widely recognized to reduce further failures.

In its most simplistic form, a structure is "any assemblage of materials which is intended to sustain loads" [1]. Typically, if an engineering structure fails it will result in loss of life or at the very least significant injury. For this reason, a great deal of effort and work goes into the design of a structure so it can properly sustain prescribed loadings. However, failures still occur. Structural failure can be induced by a wide
variety of events such as, deterioration of the structure over time (corrosion), sudden impact damage (blast), natural events (earthquake, tornado, and typhoon) and improper initial design. [1]

The concept of robustness, redundancy and static indeterminacy are key in many design philosophies and widely recognized as an important aspect in structural engineering. However, finding a consistent definition of the redundancy and robustness can be challenging. For example, the definition of redundancy may be provided in terms of collapse load, number of plastic hinges, the probability of system failure, etc. Others tend to use the term redundancy and static indeterminacy interchangeably. It has been that the degree of static indeterminacy does not correlate to structural redundancy. Structures with lower degree of static indeterminacy can often times have greater redundancy than their higher degree counter parts. This is due to the fact the redundancy relies on a wide variety of factors like, member size, material properties, structural topology, loading sequence and applied loading. Generally, redundancy is the ability of a structural system to redistribute loads among members that cannot be sustained by another member due to damage. Whereas, robustness is the ability of a structural system to sustain a specific amount of damage not disproportionate to the cause of the damage itself. [2]

In this thesis, the effects of prescribed damage scenarios on several structural systems are investigated. The structural systems investigated are then damaged from progressive deterioration of member material properties. The amount of damage is
prescribed by a damage index associated with specific patterns of cross-sectional deterioration. Once the damage is defined, structural performance will be evaluated and compared to the original intact structure.

### 1.2 Problem statement

The methods of finding optimum design solutions for structural systems can be very cumbersome to solve by hand, due to the large number of design variables present in the problem. The designer must decide which parameters are important for their current problem. Typically, in structural optimization problems, minimum weight is the desired search criterion. Optimal design of structural systems are normally limited by several constraints such as choice of material, feasible strength, displacements, deflection, size constraints, load cases, support conditions, and beam-column behavior. This research utilizes Harmony Search optimization algorithm for optimizing structural systems member sizes with both discrete and continuous design variables.

### 1.3 Objectives

The depth of this thesis is to develop a computer model that utilizes the harmony search algorithm for optimization of steel structures. Strength constraints from the AISC Load and Resistant Factor Design specification will be used along with, displacement, deflection and member size constraints. This model will then be adapted for damage tolerant optimization. Lastly, a brief section will discuss the correlation between damage and reliability.

### 1.4 Methodology

In order to achieve the objectives presented in section 1.3, the following approach was taken:

1. Perform a literature review of previous research related to optimization and damage tolerance
2. Develop a suitable harmony search algorithm code
3. Test the developed code to benchmark examples
4. Implement damage tolerance constraints into the harmony search routine
5. Compare optimization results
6. Draw conclusions from the results

### 1.5 Organization of thesis

Chapter 2 presents the literature review of the AISC code, beam-column connections, behavior of semi-rigid connections, optimization and uncertainty in engineering. Chapter 3 presents an explanation of optimization and various techniques followed by a detailed overview of harmony search optimization. Chapter 4 covers the topic of risk and reliability in engineering. Chapter 5 presents the modeling of structural systems. Chapter 6 presents design examples of the topics covered in previous chapters. Chapter 7 presents a conclusion and recommendations for future work.

## Chapter 2: Literature Review

### 2.1 Introduction

This chapter discusses background information that is relevant to the current study. As discussed in Chapter 1, the goal of this research is to provide a computer model capable of optimizing steel structures. The information presented in this chapter provides a foundation for achieving this goal. The following sections discuss the AISC code specifications, connections, optimization, harmony search algorithm, reliability and structural damage.

### 2.2 AISC-LRFD specification of connections

Connections are the components that hold a steel structure together. Typically, structural connections are bolted or welded together in different configurations depending on the system. According to AISC, there are several types of steel connections: simple framing (unrestrained), rigid-frame (fully restrained), semi-rigid framing (partially restrained) and truss connections. [3]

### 2.2.1 Truss connections

In truss connections, only axial forces are transferred through the connection. They allow a full range of rotation and are considered one of the most simplistic connections. [3]

### 2.2.2 Simple connections

A simple connection, also known as a shear connection, can transmit shear and a negligible moment force through the connection. The connection allows unrestrained
relative rotation between the framing elements and shall have sufficient rotation capacity to accommodate the required rotation determined by analysis. Inelastic rotation of the connection is acceptable. [3]

### 2.2.3 Moment connections

Moment connections, unlike simple connections are capable of transmitting moment forces across the connection. The LRFD specification for structural steel buildings classifies two types of moment connections: fully restrained (FR) and partially restrained (PR). When connection restraint is considered strength, stiffness and ductility characteristics must be included in the analysis and design of the structural system. [3]

1. Fully-Restrained (FR) Moment Connections transfer the moment force while allowing a negligible amount of rotation between members. In analysis, this may be assumed as zero rotation between members or fully rigid connections. The connection shall have sufficient strength and stiffness to maintain the original angle between members at the strength limit states. They are particularly useful when a framing system needs to provide more flexural resistance and reduce lateral deflections. [3]
2. Partially-Restrained (PR) Moment Connections transfer the moment force while allowing rotation between members. They have insufficient rigidity to maintain the original angle between the column and beam. In analysis, force-deformation response characteristics of the connection shall be included. [3]


Figure 1 - Connection Rotation [6]

### 2.3 Types of connections

### 2.3.1 Single web angle

The single web angle connection is shown in figure 2 and consists of an angle connecting the web of the beam to the column flange. Number of bolts, angle thickness, web thickness and column thickness, influence the connections behavior. [4]


Figure 2 - Single web angle [6]

### 2.3.2 Double web angle

The double web angle connection shown in figure 3 and consists of two angles connecting the web of the beam to the column flange. Number of bolts, angle thickness, web thickness and column thickness, influence the connections behavior. [4]


Figure 3 - Double web angle [6]

### 2.3.3 Header plate

The header plate connection is shown in figure 4 and consists of an end plate that's length is less than the depth of the beam, welded to the beam and bolted to the column. The behavior of this connection is influenced by plate thickness, plate depth and beamweb thickness. This connection performs similar to the double web angle connection.


Figure 4 - Header plate [6]
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### 2.3.4 Top and seat angles

The top and seat angle shown in figure 5 consists of two angles that are connected to the top and bottom flanges of the beam then connected to the flange of the column. The top angle is used for lateral stability and is not considered to carry gravity loading. The bottom or seat angle only transfers vertical loading and provides an insignificant amount of moment restraint. The number of bolts and angle thickness influence behavior. [4]


Figure 5 - Top and set angle [6]

### 2.3.5 Top and seat angles with double web angles

The top and seat angle with double web angles is a combination of the top and seat angle connection and double web angle connection as seen in figure 6. Depth and thickness of the angles, column flange or web thickness and gauge distance of bolts in the vertical angles govern this connections behavior. Plate thickness, column flange thickness and moment arm for column flange bolts influence the behavior of this connection. [4]


Figure 6-Top and seat with double web angle [6]

### 2.3.6 Extended end plate without column stiffeners

The extended end plate connection shown in figure 7 is comprised of a plate welded to the end of the beam then fastened to the column. The plate extends past both the tension and compression flanges of the beam. Plate thickness, column flange thickness and moment arm for column flange bolts influence connection behavior. [4]


Figure 7 - Extended end plate [6]

### 2.3.7 Extended end plate with column stiffeners

The extended end plate with column stiffeners seen in figure 8 is the same configuration as the previous connection only with the addition of column stiffeners. [4]


Figure 8 - Extended end plate with stiffeners [6]

### 2.3.8 T-Stub

The T-stub connection shown in figure 9 is similar to the top and seat connection except tee sections replaces the angles. This configuration provides a very rigid joint. T-stub thickness and width influence the behavior of this connection. [4]


Figure 9 - T-stub [6]

### 2.4 Behavior and modeling of semi-rigid connections

A beam-column connection is subjected to axial and shear forces along with bending and torsion moments. When working with planar frames, the torsion moments
are often neglected. Axial and shear deformations are typically neglected because they are small relative to bending deformation of most connections. This leaves only the rotational deformation of the connection to be considered in semi-rigid connection framing. A semi-rigid connection is able to rotate through an angle $\theta_{\mathrm{r}}$ due to an applied moment M previously shown in figure 1 . The angle $\theta_{\mathrm{r}}$ is the relative rotation of the beam and the column taken at the connection.[4]

Several connections were experimentally tested by Frye and Morris to show the rotation-moment relationship. The connections moment-rotation behavior is non-linear in nature and falls between fully fixed and ideally pinned. The relationship of several types of connections is shown in figure 10. [5]


Figure 10 - Moment rotation curves [5]

### 2.5 Mathematical modeling of semi-rigid connections

The most precise and dependable knowledge of beam-column connection behavior is found through experimental testing, but these test can be very complex and expensive and are not practical for design practice. To take in account connection behavior in structural analysis, connections are typically represented by mathematical models representing rotation-moment relationships. The non-linear behavior of a connection is difficult to represent exactly by mathematical representation and the models used are approximates due to simplifications.

### 2.5.1 Linear model

The most simplistic connection model is single-stiffness linear model proposed by Batho, Rathbun and Baker with the following expression: [6]

$$
\mathrm{M}=\mathrm{R} * \emptyset_{\mathrm{r}}
$$

where $M$ represents the connection moment and $R$ and $\theta$ represent the stiffness and rotation respectively.

### 2.5.2 Polynomial model

The linear model is an over simplification of connection behavior and does not represent the true behavior of a connection. Polynomial models were proposed to provide a more accurate representation of connection behavior. Frye and Morris used an odd power polynomial model to represent the moment-rotation curve as follows: [6]

$$
\emptyset=\mathrm{C}_{1}(\mathrm{KM})+\mathrm{C}_{2}(\mathrm{KM})^{3}+\mathrm{C}_{3}(\mathrm{KM})^{5}
$$

where $\theta$ is the connection rotation and $M$ is the moment acting on the connection. The variable K is the standardization factor determined by the connection type and geometry and $\mathrm{C} 1, \mathrm{C} 2$ and C 3 are curve-fitting constants obtained by using the least squares method. These constants for various connection types can be seen in table 1. [5]
Single web-angle connection

Table 1 - Moment rotation curve fitting equations [6]

### 2.5.4 Three-parameter power model

Chen and Kishi adopted the power model to represent the rotation-moment relationship for beam-column connections as shown:

$$
\begin{gathered}
M=\frac{R_{k i}{ }^{*} \emptyset}{\left\{1+\left[\varnothing / \emptyset_{0}\right]^{n}\right\}^{1 / n}} \\
R_{k i}=\frac{d M}{d \varnothing}=\frac{R_{k i}}{\left\{1+\left[\varnothing / \emptyset_{0}\right]^{n}\right\}^{(n+1) / n}} \\
\emptyset=\frac{M}{R_{k i}\left(1+\left[\mathrm{M} / \mathrm{M}_{\mathrm{u}}\right]^{n}\right\}^{1 / n}}
\end{gathered}
$$

where $\mathrm{R}_{\mathrm{ki}}$ representes the initial stiffness, $\mathrm{M}_{\mathrm{u}}$ is the ultimate moment capacity and n is shape parameter.[6]

### 2.6 Optimization of steel structure

As today's world continues to increase in population with world resources declining the need for economical designs is at the forefront for structural engineers. More structures are needed for living and production than ever before which is why these structures need to be designed using the minimum amount of material available. Due to this need, optimization algorithms prove to be a useful tool when designing steel structures. These algorithms can be implemented while staying within design constraints specified from steel design code and search for a minimum weight or cost structure. Formulation of these optimization algorithms is through mathematical models
with discrete design variables. The reason for discrete design variables is so the design model can adopt standard steel sections commonly used in practice.

Due to the complexity of structural optimization problems, heuristic search optimization methods have been the preferred choice for designers. Genetic algorithms, simulated annealing and ant colony optimization are some of the more popular heuristic search methods used in present optimization problems. These methods are easily adaptable to structural engineering optimization problems.

Several papers have focused on the optimization of steel structures.
Pezeshk et al.., (2000) [7], researched the design of nonlinear framed structures using genetic optimization. The paper presented a genetic algorithm approach for optimum design of 2D frames using discrete structural elements. The designs were in compliance with the AISC-LRFD (1994) code. Both linear and geometrically nonlinear analysis were performed to learn how $\mathrm{P}-\Delta$ effects influenced optimal designs. It was concluded that the $\mathrm{P}-\Delta$ effects did not significantly influence the optimal designs, but in some cases, it could yield a better design. In addition, it was found that the proposed optimization approach was effective optimization technique.

Hayalioglu et al.., (2001) [8], researched optimum load and resistance factor design of steel space frames using genetic algorithm. The paper presented a genetic algorithm to design moment-resisting space frames subjected to AISC-LRFD specifications for minimum weight. They utilized standard steel sections from AISC wide-flange (W) shapes. The proposed frame was subjected to wind loading in
accordance to the Uniform Building Code (UBC). Comparisons between AISC-ASD and AISC-LRFD designs were made and showed that the former code resulted in lighter structures for the presented examples.

Hayalioglu and Degertekin (2005) [9], researched minimum cost design of steel frames with semi-ridged connections and column bases via genetic optimization. The optimization algorithm obtained the minimum total cost which comprised of total member and connection costs by selecting suitable sections from the AISC wide-flange (W) shapes. Displacement, stress and size constraints in accordance to the AISC-LRFD code were imposed on the frame. Comparisons were made between AISC-ASD and AISC-LRFD and the former code provided lighter structures. They also compared semirigid connections to rigid connections and found that reducing connection stiffness caused an increase in both frame cost and sway. The reason for these increases is the more flexible frame the larger the displacements which was compensated by increasing the member size to stay within code constraints.

Lee and Geem (2005) [10], presented a structural optimization method based on harmony search meta-heuristic algorithm. The algorithm was conceptualized using the musical process of Jazz musicians searching for a perfect stat of harmony. The advantage of HS is that unlike other optimization methods it does not require initial values and uses a random search routine instead of a gradient search. Several structural truss examples were present in the study to show the effectiveness and robustness of the
new approach. The findings showed that the HS can be a powerful search and optimization method for solving structural engineering problems.

Saka, (2008) [11], researched optimum design of steel sway frames using harmony search algorithm to British Standard BS5950. The optimum design algorithm developed imposed behavior and performance constraints in accordance to BS5950. The optimization routine used standard sections from the Universal beam and column sections of the Britich Code. Optimization results obtained from the harmony search algorithm were compared to genetic algorithms and produced lighter results.

### 2.7 Harmony search algorithm in structural engineering

Harmony search (HS) is a relatively new meta-heuristic search algorithm developed by Geem et al. [10]. The original purpose for the algorithm was for solving combinatorial optimization problems in applied mathematics, but it can be adapted to a wide variety of optimization problems. HS has been applied to a range of civil and structural engineering problems such as the optimization of trusses, frames, dams, etc.

### 2.8 Uncertainty and damage in structural engineering

In its most simplistic form, a structure has been defined as "any assemblage of materials which is intended to sustain loads". Typically, if an engineering structure fails it will result in loss of life or at the very least significant injury. For this reason, a great deal of effort and work goes into the design of a structure so it can properly sustain prescribed loadings. However, failures still occur. Structural failure can be induced by a
wide variety of events such as, deterioration of the structure over time (corrosion), sudden impact damage (blast), natural events (earthquake, tornado, and typhoon) and improper initial design. [1]

The concept of robustness, redundancy and static indeterminacy are key in many design philosophies and widely recognized as an important aspect in structural engineering. However, finding a consistent definition of the redundancy and robustness can be challenging. For example, the definition of redundancy may be provided in terms of collapse load, number of plastic hinges, the probability of system failure, etc. Others tend to use the term redundancy and static indeterminacy interchangeably. It has been presented that the degree of static indeterminacy does not correlate to structural redundancy. Structures with lower degree of static indeterminacy can often times have greater redundancy than their higher degree counter parts. This is due to the fact the redundancy relies on a wide variety of factors like, member size, material properties, structural topology, loading sequence and applied loading. Generally, redundancy is the ability of a structural system to redistribute loads among members that cannot be sustained by another member due to damage. Whereas, robustness is the ability of a structural system to sustain a specific amount of damage not disproportionate to the cause of the damage itself. [13]

All aspects of life come with uncertainty and the same holds true for all sectors of engineering. Structural engineers make many decisions during the design and construction of structures. Many of these decisions are made with uncertainty, but not
often considered due to the uncertainty being accounted for in design codes. Uncertainties arise in many aspects of structural engineering in the form of nominal capacities, resistance factors, design loads and load factors. Allowable stress design utilizes a safety factor to handle these uncertainties whereas load and resistance factor design has multiple factors. However, not all uncertainty can be accounted for in the design code. Structural engineers have to be aware of the uncertainty present in their calculations and be able to account for it accordingly.

### 2.10 Concluding remarks

Based on the study of several papers, it was concluded that the new meta-heuristics algorithm harmony search proved to be a powerful tool for optimization of structural systems. Many of the publications reviewed showed the Frye-Morris polynomial model provides an accurate representation for moment-rotation connection behavior. Lastly, it was found that the extended end plate connection is a popular connection used in steel structures and optimization routines.

## Chapter 3: Optimization

### 3.1 Introduction

As previously discussed in chapters 1 and 2, mathematical optimization, simply is the process of making something the best it can possibly be. Traditionally, optimization is performed using calculus-based methods such as numerical linear and nonlinear programming methods. These methods require substantial gradient information and can be sensitive to starting points. They are ideal for obtaining global optimum points in relatively simple models. However, real-world engineering problems tend to be very complex in nature and prove hard to solve using traditional methods. More than one optimal point may be present in these complex problems and the results would be very sensitive to the selected starting point. The optimal solution may not necessarily be the global optimum for the problem. In addition to these issues, objective functions and constraints can have multiple or sharp peaks resulting in difficult or unstable gradient computations. The drawbacks of traditional techniques led to the need of other optimization methods. Researchers utilized meta-heuristic algorithms based on simulations to solve these complex problems. These algorithms are typically based around natural phenomena and each have a unique set of rules and randomness intrinsically built in. The following sections provide a brief overview of some of the more popular meta-heuristic algorithms followed by an in depth explanation of the harmony search method.

### 3.2 Heuristic optimization techniques

Heuristic comes from the Greek work heuriskein, which means to discover. In optimization, it refers to solution strategy by trial and error to produce a reasonable solution to complex optimization problems. Due to the complexity of some problems it would be infeasible to search for all possible solutions, the aim of this strategy is to find an acceptable solution in a reasonable amount of time. There is no way to know if the best solution can be found, nor will the algorithm work or why. A heuristic algorithm is an efficient and practical approach that has been shown to provide good results, but no guarantee of optimality.

Heuristic algorithms typically fall into three broad categories; simulated annealing, traditional genetic algorithm and evolutionary algorithms. The last two categories are very similar but have slight differences in the specifics of the algorithms.

### 3.2.1 Genetic algorithm (GA)

Genetic algorithms mimic the process of biological evolution in order to solve problems and to model evolutionary systems. The foundation for GAs revolves around the premise that over many generations, natural populations evolve according to the principles of natural selection, survival of the fittest. By replicating this process GAs are able to "evolve" solutions to real world problems. The main goal of GAs is the survival of robust solutions and elimination of the weak solutions in a population. GAs were first proposed by John Holland in the 1960s and further developed by Holland and
his students [14]. The procedure for the genetic algorithm can be time consuming and the optimum solution may not be global ones, but they are feasible both mathematically and practically.

### 3.2.2 Simulated annealing (SA)

Simulated annealing (SA) is an accepted local-search technique, which utilizes the analogy between the way metals cool and freeze into a minimum energy crystalline structure (the annealing process). SA approaches the optimization problem by navigating the search space iteratively stepping from one solution to another solution. It begins at a "high" temperature, which enables it to have wide range of solutions so it can move freely around the solution space. As the temperature declines, it will settle into a relatively small range, ultimately giving an optimal solution. It was developed in 1983 to deal with highly nonlinear problems by Metropolis et al.. [15], and proposed by Kirkpatrick et al.. for optimization.

### 3.2.3 Ant colony optimization algorithm (ACO)

Ant colony optimization (ACO) is an algorithm based off ant methodology for finding food, and it is used to solve discrete optimization problems. The optimization problem can be transformed into a problem of finding the best path on a graph. The "ants" incrementally build solutions by moving among the graph. It utilizes several artificial characteristics such as memory, visibility and discrete time to come to an optimum solution. Dorigo et al.. was the first to utilize this method for optimization problems [16].

### 3.2.4 Harmony search optimization algorithm (HS)

Harmony search (HS) is a meta-heuristic algorithm that Geem et al.. developed in 2001 and makes use of the analogy between the performance of musicians and searching for optimal solutions. When musicians, play a song, he/she selects musical notes to give the best overall harmony $[10,12]$. The optimization solution vector is analogous to the harmony created by the musicians, whereas the musician's improvisations are analogous to the optimization search schemes. HS algorithm, unlike previous mentioned methods, does not require initial values for the decision variables. It utilizes a stochastic random search and has light mathematical requirements so it can easily be adapted to a wide range of optimization problems [10,12].

### 3.3 Basic of harmony search algorithm

The definition of harmony is the combination of simultaneously sounded musical notes to produce chords and chord progressions having a pleasing effect. Do, Re, Mi, Fa, Sol, La , and Si are notes, which represent a specific singular sound. HS algorithm imitates musical improvisation process where the musicians try to find a better harmony. Musicians are always striving to attain the best harmony, which can be accomplished through numerous practices of changing the notes that are played. Figure 11 gives a visual representation of the analogy between music and mathematical representation.

### 3.4 Harmony search optimization algorithm in structural engineering

Figure 12 illustrates the analogy once again in terms of a steel frame design. As explained by Lee and Geem, harmony memory (HM) is the most crucial part of the HS
methodology. Geem's inspiration for HS was musically driven, in particular, the improvisation of jazz music. The method jazz musicians use to select their notes can be broken down into three different categories; play from memory, play from memory with a slightly different pitch, or randomly play another note. Utilization of these three processes makes up the core of the harmony search algorithm [10,12].

Many jazz performances comprise of several musicians each playing a different instrument, such as a guitarist, saxophonist and a pianist. Each member has a range of pitches they are capable of playing; guitarist [Do, Re, Mi]; saxophonist [Mi, Fa, Sol]; pianist [Sol, La, Si]. Each one is capable of playing any of their available pitches. Consider the following notes are played: guitarist Do; saxophonist Mi; pianist Sol. This would result in a harmony of [Do, Mi, Sol] [10,12].


Figure 11 - Harmony search analogy [12]


Figure 12 - Harmony search and steel frames [12]

### 3.4.1 Initialize the harmony search parameters

The HS algorithm parameters are selected in the first step. They are problem dependent and can be adjusted accordingly. These parameters are as followed:

- Harmony Memory Consideration Rate (HMCR).
- Harmony Memory Size (HMS).
- Pitch Adjustment Rate (PAR).
- Number of Improvisations (NI).


### 3.4.2 Initialize harmony memory

In the second step, the harmony memory (HM) matrix is randomly generated with design variables. Each row of the harmony memory matrix contains the values of design variables which were randomly selected from feasible solutions. The matrix has N columns where N represents the total number of design variables and it has HMS rows, which was previously selected. The harmony memory matrix can be seen in equation $6[10,12]$.

$$
\left.\left[\begin{array}{ccccc}
x_{1}^{1} & x_{2}^{1} & \cdots & x_{N-1}^{1} & x_{N}^{1}  \tag{6}\\
x_{1}^{2} & x_{2}^{2} & \cdots & x_{N-1}^{2} & x_{N}^{2} \\
\vdots & \vdots & \ddots & \vdots & \vdots \\
x_{1}^{H M S-1} & x_{2}^{H M S-1} & \cdots & x_{N-1}^{H M S-1} & x_{N}^{H M S-1} \\
x_{1}^{H M S} & x_{2}^{H M S} & \cdots & x_{N-1}^{H M S} & x_{N}^{H M S}
\end{array}\right] \rightarrow \begin{array}{l}
\rightarrow \\
\rightarrow \\
\rightarrow
\end{array}\right) \quad \begin{aligned}
& \rightarrow \\
& \rightarrow
\end{aligned} x^{H M S}\left(x^{2}\right)
$$

### 3.4.3 Improvise a new harmony

In the third step, a new harmony vector, $x^{\prime}=\left(x_{1}^{\prime}, x_{2}^{\prime}, \cdots, x_{N}^{\prime}\right)$ is improvised. There are three rules to choose a value for each decision variable: memory consideration (HMCR), pitch adjustment (PAR) and random selection (RN). In harmony memory considering rate, the value of the first decision variable can be chosen from any discrete or continuous value in the specified HM range with the probability of HMCR which varies between 0 and 1 . Values of the other decision variables can be chosen in the same manner. However, there is still a chance where the new value can be randomly chosen from the entire set possible values with the probability of (1-HMCR) [10,12].

$$
x_{i}^{\prime} \leftarrow\left\{\begin{array}{r}
x_{i}^{\prime} \in\left\{x_{i}^{1}, x_{i}^{2}, \cdots, x_{i}^{H M S}\right\} \text { w.p. HMCR }  \tag{7}\\
x_{i}^{\prime} \in X_{i} \text { w.p. }(1-H M C R)
\end{array}\right.
$$

Any component of the new harmony vector, whose value was chosen from the HM, is then examined to determine whether it should be pitch-adjusted. This operation uses pitch adjusting parameter (PAR) that sets the rate of pitch-adjustment decision as follows [10,12]:

$$
x_{i}^{\prime} \leftarrow\left\{\begin{array}{c}
Y E S \text { w.p. PAR }  \tag{8}\\
N O \text { w.p. }(1-P A R)
\end{array}\right.
$$

If the pitch adjustment decision for $x_{i}^{\prime}$ is yes, $x_{i}^{\prime}$ is replaced with $x_{i}(k)$ (the $k^{\text {th }}$ element in $X_{i}$ ), and the pitch-adjusted value of $x_{i}(k)$ becomes

$$
\begin{align*}
& x_{i}^{\prime} \leftarrow x_{i}(k+c) \text { for discrete design variables }  \tag{9}\\
& x_{i}^{\prime} \leftarrow x_{i}^{\prime}+\alpha \text { for continuous design variables }
\end{align*}
$$

The algorithm chooses a value form a neighboring index m with the same probability [10,12].

### 3.4.4 Update the harmony memory

If the new harmony $x^{\prime}=\left(x_{1}^{\prime}, x_{2}^{\prime}, \cdots, x_{N}^{\prime}\right)$ is better than the worst harmony in the HM in terms of objective function value, the new harmony is included in the HM and the existing worst harmony is excluded from the HM [10,12].

### 3.4.5 Termination criteria

In the final step, the computation is terminated when the termination criterion is satisfied, typically a prescribed maximum number of iterations. Otherwise, Steps 3 and 4 are repeated until the termination criteria have been met $[10,12]$

### 3.4.6 Harmony search flow chart

| Flow Chart Legend |  |
| :--- | :--- |
| $A_{i}:$ Descrete size variables $(\mathrm{i}=1,2, \ldots, n)$ | PVS( ${ }^{*}$ ): Possible value set for $\mathrm{A}_{\mathrm{i}}$ |
| HMCR: Harmony memory considering rate | nPVS: Number of possible value sets for $\mathrm{A}_{\mathrm{i}}$ |
| PAR: Pitch adjustment rate | NDHV( ${ }^{*}$ ): New discrete harmony vector |
| HMS: Harmony memory size | E1: Memory considerations process |
| HM(*,*): Harmony memory | E2: Pitch adjustment process |
| ran: Random number in the range 0.0 to 1.0 | E3: Randomization process |

Table 2 - Harmony search flow chart legend


Figure 13 - Harmony search flow chart [12]

### 3.5 Comparison between harmony search and other optimization techniques

### 3.5.1 Harmony search example: 10-bar planar truss



Figure 14-10 bar truss configuration
The cantilever truss, shown in figure 14, has been previously analyzed using various mathematical optimization methods. The material density for the truss was $0.1 \mathrm{lb} / \mathrm{in}^{3}$ and the modulus of elasticity was $10,000 \mathrm{ksi}$. Stress limitations were $+/-25 \mathrm{ksi}$, and displacements at each node were limited to $+/-2.0$ inches in both x and y directions. The loading case used was with two single loads of $\mathrm{Q}=100 \mathrm{kips}$. The minimum crosssectional area of the members was $0.1 \mathrm{in}^{2}$ and there were no maximum area limitations. Table 1 provides the data from this thesis along with several other publications for the same configuration. The one of most interest to this paper would be the findings of Kang Seok Lee [10]. Lee utilized a basic harmony search to find a minimum weight of
5057.88 lbs , which is the lowest of all the studies found in this thesis. The same harmony search methodology was ran using MATLAB for this thesis, it performed better than most of the other reports, but was unable to replicate the same results Lee found. The results were very close with only a $0.13 \%$ difference. Different formulation of the constraints and the constraint handling methods could be the reason for the discrepancy. The MATLAB code ran for this thesis utilized a static penalty function in the handling of constraints. The harmony search optimization code used for this problem provided an adequate answer and will be used for the damage tolerant optimization routines.

|  |  | $\begin{aligned} & \stackrel{\text { O}}{\underset{\sim}{c}} \end{aligned}$ | $\begin{aligned} & \stackrel{\rightharpoonup}{\mathrm{O}} \\ & \underset{\sim}{\prime} \end{aligned}$ | $\stackrel{\tilde{\sim}}{\stackrel{\sim}{m}}$ | $\stackrel{\rightharpoonup}{0}$ | $\stackrel{\rightharpoonup}{0}$ | $\stackrel{\rightharpoonup}{0}$ | $\underset{\infty}{\infty}$ | $\underset{\sim}{\star}$ | $\begin{aligned} & \text { ợ } \\ & \underset{\sim}{2} \end{aligned}$ | $\stackrel{\rightharpoonup}{\circ}$ | ¢\% |
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|  |  | $\stackrel{\stackrel{\rightharpoonup}{\dot{N}}}{ }$ | $\stackrel{N}{\underset{\sim}{j}}$ | $\begin{gathered} \hat{N} \\ \stackrel{n}{0} \end{gathered}$ | $\stackrel{\hat{o}}{\dot{\circ}}$ | $\stackrel{\rightharpoonup}{0}$ | $\stackrel{\sim}{\circ}$ | $\underset{\sim}{\sim}$ | $\stackrel{\underset{\sim}{i}}{\underset{\sim}{n}}$ | $\underset{\sim}{i}$ | $\stackrel{\sim}{\circ}$ | ¢ |
|  |  | $\stackrel{\underset{\sim}{\underset{\sim}{*}}}{ }$ | $\begin{aligned} & \underset{\sim}{7} \\ & \underset{\sim}{2} \end{aligned}$ | $\begin{gathered} \text { ָै } \\ \stackrel{\sim}{6} \end{gathered}$ | $\stackrel{n}{0}$ | $\stackrel{\rightharpoonup}{0}$ | $\stackrel{\rightharpoonup}{0}$ | $\stackrel{\circ}{\infty}$ | $\begin{aligned} & \infty \\ & \stackrel{\rightharpoonup}{i} \end{aligned}$ | $\begin{aligned} & \infty \\ & \stackrel{\rightharpoonup}{i} \end{aligned}$ | $\stackrel{\rightharpoonup}{0}$ | - |
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|  |  | $\begin{aligned} & \text { O} \\ & \dot{\sim} \end{aligned}$ |  | $\stackrel{\infty}{\circ}$ | $\stackrel{?}{0}$ | $\stackrel{\rightharpoonup}{0}$ | Hivi | $\stackrel{\underset{\sim}{\mathrm{N}}}{ }$ | $\stackrel{\text { ®ে }}{\stackrel{\circ}{\circ}}$ | $\underset{\sim}{\underset{\sim}{\sim}}$ | $\stackrel{\rightharpoonup}{\circ}$ | ¢ ¢ in |
|  |  | な | ¢ | < | * | 8 | < | < | ¢ | 8 | 8 |  |

Table 3-10 bar truss optimization comparison [12]

### 3.6 Damage tolerant optimization

### 3.6.1 General mathematical formulation

Find:

$$
\begin{equation*}
\text { Design Variables } X=\left[X_{1}, X_{2}, \ldots, X_{n}\right]^{T} \in S \tag{10}
\end{equation*}
$$

That minimizes:

Objective Function: $f(X)=\left[f_{1}(X), f_{2}(X), \ldots, f_{i}(X), \ldots, f_{m}(X)\right]^{T}$
The feasible set S belongs to an n -space determined by a set of equality and inequality conditions:

$$
\begin{gather*}
\text { Equality } g(X)=0  \tag{12}\\
\text { Inequality } h(X)<0 \text { or }>0
\end{gather*}
$$

For damage tolerant optimization the objective function is composed of:

| Weight | $(W)$ | 14 |
| :--- | :---: | :---: |
| Intact Capacity | $\left(C_{U}\right)$ | 15 |
| Residual Capacity | $\left(C_{R}\right)$ | 16 |
| Displacements | $(\Delta)$ | 17 |

Damage tolerant objective function:

$$
\begin{equation*}
f(X)=\left[W, C_{U}, C_{R}, \Delta\right]^{T} \tag{18}
\end{equation*}
$$

The design variable vector for a damage tolerant truss system is made up of member areas

$$
\begin{equation*}
X=\left[A_{1}, A_{2}, \ldots, A_{n}\right]^{T} \tag{19}
\end{equation*}
$$

Bounds can be imposed on the cross-sectional areas resulting in the following feasible set for damage tolerant optimization

$$
\begin{equation*}
S=\left\{X \in R^{n}: A_{i, \min } \leq A_{i} \leq A_{i, \max } i=1,2, \ldots, n\right\} \tag{20}
\end{equation*}
$$

The above multi-objective problem can be transformed into a series of single objective minimization problems using the $\in$-constraint method.

$$
\begin{equation*}
\min V \tag{21}
\end{equation*}
$$

Satisfying

$$
\begin{array}{cc}
C_{U} \geq C_{U}^{0} & 22 \\
C_{R} \geq C_{R}^{0} & 23 \\
\Delta_{j, \text { max }} \leq \Delta_{j}^{0}, \quad j=1,2, \ldots, m & 25 \\
A_{i, \text { min }} \leq A_{i} \leq A_{i, \text { max }}, \quad i=1,2, \ldots, n
\end{array}
$$

The required residual capacity $C_{R}^{0}$ can be varied to cover the entire solution set. For this study we were focused on the influence of residual capacity requirements on the optimization results.

Frangopol and Klisinksi proposed a three load level checking design; nominal load $\left(Q_{N}\right)$, ultimate load $\left(Q_{U}\right)$, and the residual load $\left(Q_{R}\right)$. Nominal and ultimate loads are used to check the serviceability and ultimate capacity requirements, respectively. The residual load is used to check the residual capacity requirement under potential future damage conditions to the structural system. Damage conditions can be represented by reductions in stiffness of members, complete removal of members, or combination of these.

In this thesis, damage conditions are assessed by complete removal of a structural member. The removal of a structural member creates a damaged structural system that will have a different performance from the original intact structure. This process can be repeated for every member since it is assumed all damage conditions are equally
probable. Once the member is removed, the damaged structural system is analyzed to find out its load capacity. The damage system that provides the lowest capacity will define the residual capacity of the intact structural system, as follows:

$$
\begin{equation*}
C_{R}=\min \left(C_{1}, C_{2}, \ldots, C_{i}, \ldots, C_{n}\right) \tag{17}
\end{equation*}
$$

where $C_{i}$ is the capacity of the structure having member $i$ removed from the system. This approach is only valid for statically indeterminate structures. Statically determinate systems require the contribution of all members to function. If a member was removed from a determinate system it would collapse. Statically indeterminate structures are inherently redundant. This allows the possible removal of one or more members from the system without the potential of collapse. Some systems may still have critical members present that cannot be removed without resulting in a system collapse. The correct indeterminate configuration must be chosen for the system to allow the consideration of residual capacity in structural optimization.

## Chapter 4: Structural Reliability

### 4.1 Introduction

Structural reliability revolves around the uncertainties associated with the design of structures and assessing the safety of the structure. Reliability is a relatively new technique in the structural engineering field that became prominent in the 1980's. It was first implemented in the AISC code in the form of the load resistance factor design (LRFD) in 1986 as an alternative to the existing allowable stress design (ASD). Most engineering problems are solved under the assumption of deterministic values (i.e. no randomness is involved in the value being used). In dealing with real world problems, uncertainties are unavoidable. Engineers must recognize the presence of uncertainty and account for it appropriately. Uncertainty can be classified into two expansive categories: First, those associated with natural randomness (aleatory) and second those associated with inaccuracies in human prediction and estimations (epistemic). When engineers are dealing with uncertainty, their goal is to reduce the total amount present in the current problem. The total uncertainty in a problem is the combination of both aleatory and epistemic uncertainties. Aleatoric cannot be reduced due to its intrinsic randomness, i.e., one would be hard pressed to limit the amount of earthquakes, storms, and other natural events. However, epistemic can be reduced by increasing our knowledge and providing better estimations and predictions. [17] Epistemic uncertainty in structural engineering would be material properties such as yield strength, modulus of elasticity, thickness, etc. The random behavior of the basic
strength can cause the strength of the structure to vary beyond acceptable limits. To account for this random behavior one must quantify the uncertainty, or randomness to account for this fluctuation of material properties. Uncertainty may be calculated using simulation techniques, such as Monte-Carlo simulation, which allows the values to be generated based on their statistical distribution (probability density function). Alternatively, the uncertainty can be estimating via first-order reliability method (FORM) or second-order reliability method (SORM). [17, 18]

The truss structure is defined as being made up of elements that can be in one of two states, an initial linear elastic state (safe) or a final zero-stiffness state (failure). This type of behavior is consistent with brittle material properties. For this type of structure, one can identify sequences of element failures that would lead to a structural collapse (failure event). Structural failure will occur if any failure event were to arise, i.e., the structural failure is a union of all failure sequences. This basic format allows conventional probability formulations in terms of unions and intersections to be used to represent the structure failure event. [18] An individual structural member is considered safe or reliable when the capacity of the member exceeds the demand being placed on the member. A degree of uncertainty will be associated with both the load $(\mathrm{L})$ and the resistance $(\mathrm{R})$. To understand the random nature of $L$ and $R$ the uncertainty must be quantified and evaluated. This is typically done through a series of test such as the ones performed by Galambos and Ravindra in 1978 on the properties of steel [19]. From these results, probability density functions
can be formulated to give a representation of the random properties of the material. The probability of safe performance $\left(P_{s}\right)$ can be expressed as:

$$
\begin{equation*}
P_{s}=P(R>L)=P(R-L>0)=\iint_{R>L} f_{R, L}(r, l) d r d l \tag{26}
\end{equation*}
$$

where $f_{R}(r)$ and $f_{L}(l)$ are the probability density functions of R and L and $f_{R, L}(r, l)$ is their joint probability density function. [18]

This concept can be delineated by figure 1 where the independent failure probability of both the L and the R is shown. If an incremental load value dl is considered, the probability of the load value falling into the interval dl and the strength value simultaneously exceeding the load value at that point gives the reliability of that segment $d P_{s}$ which can be expressed as:

$$
\begin{equation*}
d P_{s}=f_{L}(l) d l \int_{l}^{\infty} f_{R}(r) d r=f_{L}(l) d l\left[1-F_{R}(l)\right] \tag{27}
\end{equation*}
$$

where: $F_{R}$ represents the cumulative distribution function of R and $F_{R}(l)$ is indicated as area $A_{r}$ in figure 15 . The term $f_{L}(l) d l$ is represented by area, $A_{l}$. [18]

Since the reliability of the members involves the probability of the strength exceeding the load, the total reliability $\left(P_{S}\right)$ of the member is expressed as:

$$
\begin{equation*}
P_{s}=\int d P_{s}=\int_{-\infty}^{\infty} f_{L}(l)\left[\int_{l}^{\infty} f_{R}(r) d r\right] d l=\int_{-\infty}^{\infty} f_{L}(l)\left[1-F_{R}(l)\right] d l \tag{28}
\end{equation*}
$$

Failure is defined as the probability that the member will not survive. This means that the probability of failure $\left(P_{f}\right)$ can be expressed as:

$$
\begin{equation*}
P_{f}=1-P_{s}=1-P(R \geq L)=\int_{-\infty}^{\infty} f_{L}(l) F_{R}(l) d l \tag{29}
\end{equation*}
$$

The failure probability is often computed from the reliability index $\beta$.
where $\Phi$ is the distribution function of the standard normal variety. The reliability index graphically depicts the shortest distance from the origin to a failure surface in standard normal space. [18]


Figure 15 - Reliability diagram [18]

## Chapter 5: Modeling of steel structures

### 5.1 Introduction

One of the most critical steps in structural analysis is the modeling process of how members will relate to each other. A model, via finite element analysis software or user defined code needs to provide an accurate representation of its members and components to function properly. One of the most difficult parts of structural analysis is developing a sound and accurate representation of these members. Rarely, if ever, it is possible to model a structural system exactly as it occurs in nature, the user must make some general assumptions about how the structure will behave. This assumptions assume structural material deform according to basic mechanics of materials. The degree of accuracy typically depends several factors such as the complexity of the model, time and cost.

### 5.2 MATLAB

Structural analysis for this thesis was performed in the MATLAB computer program using the direct stiffness method. Using the stiffness method requires an understanding of the concept of kinematic degrees of freedom (DOF). The kinematic degrees of freedom of a body are those motions that describe its position relative to some arbitrary base position. For example, if we consider a point in Cartesian space we can measure its movement by three translations $u, v$, and $w$ in the $x, y$, and $z$ directions. A rigid body in space can have rotational movement as well. To consider these movements we need to measure $\theta \mathrm{x}, \theta \mathrm{y}$, and $\theta \mathrm{z}$, the rotations around the $\mathrm{x}, \mathrm{y}$, and z axes respectively, to
completely describe the motion of points on a rigid body. This assumption assumes rigid body motion. If one were to consider a deformable body, there would be an infinite number of degrees of freedom. Each point on the body could move relative to its surrounding points.

When analyzing structures, part of the challenge is to identify which degrees of freedom are to be used. If the structure is a deformable body with an infinite number of degrees of freedom, we must choose between "exact methods" and "approximate or numerical methods." The exact methods require the solution to differential equations with the appropriate boundary conditions applied. However, due to complex and irregular shapes solving these differential equations can be very difficult, if not impossible. This is why approximate methods are typically used to solve engineering problems.

Classical approximate solutions are usually based on approximating the displacement or stress fields in the body with series approximations or finite differences. This reduces the degrees of freedom from infinity to the number of coefficients in the approximating function. The accuracy of these methods depends heavily on how well the approximating function simulates the actual solution.

In finite element methods, the structure is approximated as a series of discrete elements that use various techniques to represent internal behavior associated with the element. Typically, when representing buildings, bridges, and other structures line elements are used. These are finite elements with nodes located at each end of the element. The structural degrees of freedom are all of the element degrees of freedom. This can result
in a very large number of degrees of freedom. However, with the advancements of computers large problems can be solved with minimal effort.

### 5.2.1 Modeling of truss

The forces in a truss element are completely determined if the displacements of the joints and the axial loads applied directly to the element are known. If we ignore element loads, the behavior of an entire truss can be determined if displacements of all the nodes are known; the nodal displacements are the degrees of freedom for formulating the problem. Truss members are represented by line elements that only support axial forces.

### 5.2.2 Modeling of Frame

Planar beam elements have 4 degrees of freedom. When combined with the truss elements degree of freedom it can be used to represent a frame element with 6 degrees of freedom ( 3 per node) capable of recognizing both axial and bending deformations. However, at the element level, these two basic types of response do not intact with each other as long as small deflections are considered. Bending of the element does not change the length of the member.

### 5.2.3 Stiffness method flow chart



Figure 16 - Stiffness method flow chart

### 5.2.4 Sources of Nonlinearity

In linear elastic analysis, the materials investigated are assumed linear elastic, meaning the material is unyielding and its properties unchanging. The equations of equilibrium are formulated on the geometry of the unloaded initial structural configuration. It assumes very small subsequent deformations. This approach has the ability to treat axial force, bending moments and torques as uncoupled actions in stiffness equations. Several options are available to address the issues from the linear elastic assumptions. These can be generalized into two categories, geometric nonlinearities and material nonlinearities. The first category, geometric nonlinearities continues to treat the structure as an elastic material but includes the effects of deformation and finite displacements in formulating the equations of equilibrium. The latter category, material nonlinearity considers the effect of changes in member material properties under load.

### 5.2.5 Semi-rigid connection nonlinearity

Semi-rigid connections are a source of material nonlinearities. The modeling of semirigid connections is typically handled by modifying the member stiffness matrix to account for the connection flexibility via end-fixity factors. The implementation of the concept of end-fixity factor into frame analysis can be done by multiplying the rigid member stiffness matrix $S_{i}$, by a correction matrix, $C_{e-i}$ seen in equations 31. [6]

$$
\mathrm{K}_{\mathrm{i}}^{\mathrm{SR}}=\mathrm{S}_{\mathrm{i}} * \mathrm{C}_{\mathrm{e}-\mathrm{i}}
$$

$$
\begin{aligned}
& S_{i}=\left[\begin{array}{cccccc}
\frac{\mathrm{EA}}{\mathrm{~L}} & 0 & 0 & \frac{-\mathrm{EA}}{\mathrm{~L}} & 0 & 0 \\
0 & \frac{12 \mathrm{EI}}{\mathrm{~L}^{3}} & \frac{6 \mathrm{EI}}{\mathrm{~L}^{2}} & 0 & \frac{-12 \mathrm{EI}}{\mathrm{~L}^{3}} & \frac{6 \mathrm{EI}}{\mathrm{~L}^{2}} \\
0 & \frac{6 \mathrm{EI}}{\mathrm{~L}^{2}} & \frac{4 \mathrm{Ei}}{\mathrm{~L}} & 0 & \frac{-6 \mathrm{EI}}{\mathrm{~L}^{2}} & \frac{2 \mathrm{EI}}{\mathrm{~L}} \\
\frac{-\mathrm{EA}}{\mathrm{~L}} & 0 & 0 & \frac{\mathrm{EA}}{\mathrm{~L}} & 0 & 0 \\
0 & \frac{-12 \mathrm{EI}}{\mathrm{~L}^{3}} & \frac{-6 \mathrm{EI}}{\mathrm{~L}^{2}} & 0 & \frac{12 \mathrm{EI}}{\mathrm{~L}^{3}} & \frac{-6 \mathrm{EI}}{\mathrm{~L}^{2}} \\
0 & \frac{6 \mathrm{EI}}{\mathrm{~L}^{2}} & \frac{2 \mathrm{EI}}{\mathrm{~L}} & 0 & \frac{-6 \mathrm{EI}}{\mathrm{~L}^{2}} & \frac{4 \mathrm{Ei}}{\mathrm{~L}}
\end{array}\right] \\
& \mathrm{C}_{\mathrm{e}-\mathrm{i}}=\left[\begin{array}{cccccc}
1 & 0 & 0 & 0 & 0 & 0 \\
0 & \frac{4 r_{2}-2 r_{1}+r_{1} r_{2}}{4-r_{1} r_{2}} & \frac{-2 \operatorname{Lr}_{1}\left(1-r_{2}\right)}{4-r_{1} r_{2}} & 0 & 0 & 0 \\
0 & \frac{6\left(r_{1}-r_{2}\right)}{\mathrm{L}\left(4-r_{1} r_{2}\right)} & \frac{-3 r_{1}\left(2-r_{2}\right)}{4-r_{1} r_{2}} & 0 & 0 & 0 \\
0 & 0 & 0 & 1 & 0 & 0 \\
0 & 0 & 0 & 0 & \frac{4 r_{1}-2 r_{2}+r_{1} r_{2}}{4-r_{1} r_{2}} & \frac{2 \operatorname{Lr}_{2}\left(1-r_{1}\right)}{4-r_{1} r_{2}} \\
0 & 0 & 0 & 0 & \frac{6\left(r_{1}-r_{2}\right)}{L\left(4-r_{1} r_{2}\right)} & \frac{-3 r_{2}\left(2-r_{1}\right)}{4-r_{1} r_{2}}
\end{array}\right]
\end{aligned}
$$

where end-fixity factors $\mathbf{r}_{\mathbf{1}}$ and $\mathbf{r}_{\mathbf{2}}$ are defined by:

$$
\begin{equation*}
r_{j}=\frac{1}{1+3 E I / R_{j} L} \quad(j=1,2) \tag{34}
\end{equation*}
$$

where end connection spring stiffness, $\mathrm{R}_{\mathrm{j}}$, is defined by the Frye and Morris curve fitting model in chapter 2 . To take into account the nonlinear behavior of semi-rigid connections, an iterative process is used to obtain the solution. In each iteration, the member stiffness is modified by the correction matrix with updated end-fixity factors $\mathbf{r}_{\mathbf{1}}$ and $\mathbf{r}_{2}$. [6]

### 5.2.6 Geometric nonlinearity

To account for geometric nonlinearities in structural systems a second-order elastic analysis needs to be performed. For rigid frames, the computer based second-order
elastic analysis is often done as an iterative procedure and the stiffness matrix of each member is composed of the elastic stiffness matrix and the geometrical stiffness matrix as show in equation 35: [6]

$$
\begin{gather*}
\mathrm{K}_{\mathrm{i}}=\mathrm{S}_{\mathrm{i}}+\mathrm{G}_{\mathrm{i}}  \tag{35}\\
\mathrm{G}_{\mathrm{i}}=\frac{\mathrm{N}}{\mathrm{~L}}\left[\begin{array}{cccccc}
0 & 0 & 0 & 0 & 0 & 0 \\
0 & \frac{6}{5} & \frac{\mathrm{~L}}{10} & 0 & \frac{-6}{5} & \frac{\mathrm{~L}}{10} \\
0 & \frac{\mathrm{~L}}{10} & \frac{2 \mathrm{~L}^{2}}{15} & 0 & \frac{-\mathrm{L}}{10} & \frac{-\mathrm{L}^{2}}{30} \\
0 & 0 & 0 & 0 & 0 & 0 \\
0 & \frac{-6}{5} & \frac{-\mathrm{L}}{10} & 0 & \frac{6}{5} & \frac{-\mathrm{L}}{10} \\
0 & \frac{\mathrm{~L}}{10} & \frac{-\mathrm{L}^{2}}{30} & 0 & \frac{-\mathrm{L}}{10} & \frac{2 \mathrm{~L}^{2}}{15}
\end{array}\right]
\end{gather*}
$$

The geometrical stiffness matrix can also take into account semi-rigid connections with the addition of a correction matrix, $\mathrm{C}_{\mathrm{g}-\mathrm{i}}$ as seen in the following equations. [6]

$$
\mathrm{G}_{\mathrm{i}}=\left[\begin{array}{cccccc}
0 & 0 & 0 & 0 & 0 & 0 \\
0 & 1 & 0 & 0 & 0 & 0 \\
0 & \mathrm{C} 1 & \mathrm{C} 2 & 0 & \mathrm{C} 3 & \mathrm{C} 4 \\
0 & 0 & 0 & 0 & 0 & 0 \\
0 & 0 & 0 & 0 & 1 & 0 \\
0 & \mathrm{C} 5 & \mathrm{C} 6 & 0 & \mathrm{C} 7 & \mathrm{C} 8
\end{array}\right]
$$

where

$$
\begin{aligned}
& \mathrm{C} 1=-\mathrm{C} 3=\frac{-4}{5 \mathrm{~L}\left(4-\mathrm{r}_{1} \mathrm{r}_{2}\right)^{2}}\left(8 \mathrm{r}_{1}^{2} \mathrm{r}_{2}-13 \mathrm{r}_{2}^{2} \mathrm{r}_{1}-32 \mathrm{r}_{1}^{2}-8 \mathrm{r}_{2}^{2}+25 \mathrm{r}_{1} \mathrm{r}_{2}+20\right) \\
& C 2=\frac{r_{1}}{5\left(4-r_{1} r_{2}\right)^{2}}\left(16 r_{2}^{2}+25 r_{2}^{2} r_{1}-96 r_{1} r_{2}+128 r_{1}-28 r_{2}\right) \\
& C 4=\frac{4 r_{2}}{5\left(4-r_{1} r_{2}\right)^{2}}\left(16 r_{1}^{2}-5 r_{1}^{2} r_{2}+9 r_{1} r_{2}-28 r_{1}+8 r_{2}\right) \\
& \mathrm{C} 5=-\mathrm{C} 7=\frac{-4}{5 \mathrm{~L}\left(4-\mathrm{r}_{1} \mathrm{r}_{2}\right)^{2}}\left(8 \mathrm{r}_{2}^{2} \mathrm{r}_{1}-13 \mathrm{r}_{1}^{2} \mathrm{r}_{2}-32 \mathrm{r}_{2}^{2}-8 \mathrm{r}_{1}^{2}+25 \mathrm{r}_{1} \mathrm{r}_{2}+20\right) \\
& \mathrm{C} 6=\frac{4 r_{1}}{5\left(4-r_{1} r_{2}\right)^{2}}\left(16 r_{2}^{2}-5 r_{2}^{2} r_{1}+9 r_{1} r_{2}-28 r_{2}+8 r_{1}\right)
\end{aligned}
$$

$$
\begin{equation*}
\mathrm{C} 8=\frac{r_{2}}{5\left(4-r_{1} r_{2}\right)^{2}}\left(16 r_{1}^{2}+25 r_{1}^{2} r_{2}-96 r_{1} r_{2}+128 r_{2}-28 r_{1}\right) \tag{43}
\end{equation*}
$$

### 5.4 SAP2000

SAP2000 is a general-purpose engineering software package ideal for analysis and design of structural systems. It can represent a wide range of systems from basic 2-D systems to complex 3-D structures. Modeling of the structural elements is typically handled through a graphical user interface (GUI) that allows the user to define members, loading, material properties, etc. The user also has the option to edit an input file to define the structural system.

SAP2000 was utilized to validate the MATLAB model. Both analysis methods were performed on the three story, two bay frame with semi-rigid connections shown in


Figure 17 - Frame diagram
figure 17. The material properties of the structural members were in accordance to the AISC steel design manual and the modulus of elasticity was set at $30,000 \mathrm{ksi}$ for the analysis.

To account for semi-rigid connections the moment rotation curve shown in figure 18 was used. The moment rotation curve is representative of an extended end plate connection without column stiffeners and the curve fitting constancies for this configuration can be seen in table 1 in chapter 2 . The end plate was assumed to have a thickness of $0.685^{\prime \prime}$ using $1^{\prime \prime}$ diameter bolts with a spacing between bolt groups equal to the member depths plus 6 ".


Figure 13 - Extended end plate moment rotation curve

SAP2000 does not have the capabilities to specify moment rotation interaction of connections. Instead, a rotational spring must be used at the connection points to represent the connection flexibility. For the testing analysis a secant stiffness of $6.35 \times 10^{5}(\mathrm{~K} . \mathrm{in} / \mathrm{rad})$ was used as the member partial fixity values in both computer models.

|  | Nonlinear analysis of Semi-rigid Frame |  |  |
| :---: | :---: | :---: | :---: |
|  | MATLAB | SAP2000 | Percent Diffrence |
| Upper right corner <br> displacmenet (in.) | 1.12 | 1.15 | $2.64 \%$ |
| Maximum column base <br> moment (K-in) | 919.4 | 905.61 | $1.51 \%$ |

Table 4 - Analysis comparison
The results obtained from both models were relatively similar when compared to one another. The nodal displacements of the MATLAB model were all within 1-3\% of the SAP model. Also, element bending, shear and axial loading vary from about $1-2 \%$ of each other. The results show that the MATLAB model is an acceptable representation of the structural system and can be used for the optimization process.

## Chapter 6: Design Examples

### 6.1 Damage tolerant truss

The 10-bar truss configuration
shown in figure 2 with three proportional loads Q will be analyzed for damage tolerant optimization as outlined in

Chapter 3. Geometrical, mechanical and loading


Figure 14 - Ten bar truss configuration optimization characteristic were assumed to be deterministic. The modulus of elasticity was assumed to be $\mathrm{E}=29,000 \mathrm{ksi}$. Each member was assigned its own individual cross sectional area $\mathrm{A}_{\mathrm{i}}$ as shown in figure 2. The material was assumed to be brittle with yielding stresses of +/- 25ksi. Buckling constraints were also applied to each individual member. The minimum and maximum cross-sectional areas were $\mathrm{A}_{\text {imin }}=0.1$ and $\mathrm{A}_{\text {imax }}=$ infinity. The initial cross sectional areas of the truss were set at $\mathrm{A}_{\mathrm{i}}=1$ inch.

### 6.1.2 Solving damage tolerant optimization problem

The design variables for this problem are restricted to the cross sectional areas of the structural members. The geometry of the structure and material properties are considered fixed. The cross sectional areas of the members are the design variables $A_{i}$, subjected to size constraints

$$
\begin{equation*}
A_{i, \min }<A_{i}<A_{i, \max } ; i=1, \ldots, n \tag{44}
\end{equation*}
$$

where $A_{i, \min }$ and $A_{i, \max }$ are the minimum and maximum member areas, respectively. In this problem the minimum member area is limited to $0.1 \mathrm{in}^{2}$ and the maximum area is not limited.

The objective function for the optimization problem is to minimize the volume of the structure:

$$
\begin{equation*}
V=\sum_{i=1}^{n} l_{i} a_{i} \tag{45}
\end{equation*}
$$

where $l_{i}$ is the total length of the members and $a_{i}$ is the corresponding area of the member. The volume $V$ can be multiplied by the unit weight of the structure to provide an adequate assessment of the structures cost. The following constraints must be satisfied.

Ultimate load carrying capacity requirement:

$$
\begin{equation*}
C_{U} \geq C_{U}^{0} \tag{46}
\end{equation*}
$$

where $C_{U}$ and $C_{U}^{0}$ are the actual and the required ultimate load carrying capacity of the system respectively.

Serviceability requirements:

$$
\begin{equation*}
\Delta_{i} \leq \Delta_{i}^{0}, \text { for all } i \tag{47}
\end{equation*}
$$

where $\Delta_{i}$ and $\Delta_{i}^{0}$ are the maximum and the allowable elastic displacement at section $i$, respectively. It is logical to compute the displacements under the nominal load $\mathrm{Q}_{N}$. Residual capacity requirement:

$$
\begin{equation*}
C_{R} \geq C_{R}^{0} \tag{48}
\end{equation*}
$$

where $C_{R}$ and $C_{R}^{0}$ are the actual and the required residual capacity of the system, respectively.

Reserve strength factor is defined as:

$$
\begin{equation*}
R_{1}=C_{U} / Q_{N} \tag{49}
\end{equation*}
$$

The reserve factor is a measurement of strength that compares the ultimate load to the nominal loading. The reserve strength factor, $R_{1}$, can range from value of 0 when the intact structure has no loading effect, to a value of 1.0 when the nominal load on the intact structure equals its capacity, $C_{U}$.

Residual strength factor is defined as:

$$
\begin{equation*}
R_{2}=C_{R} / C_{U} \tag{50}
\end{equation*}
$$

The residual strength factor, $R_{2}$, is used to show the strength of a structure once it is in a damage state. This value can range from 0 when the damage structure is collapsed to a theoretical value of 1.0 when the damage structure can carry the same load capacity as the intact structure. Frangopol and Klisinski show that for a given structural configuration, loading and material behavior there is always a maximum value of residual strength. This is due to the fact that the residual capacity of the structural system, $C_{R}$, cannot increase over a certain threshold without raising the ultimate capacity of the intact structure, $C_{U}$.

### 6.1.3 Optimization results

First, the behavior of the initial intact structure was investigated. The initial structure had a volume of $1165.69 \mathrm{in}^{3}$. Table 5 shows the stress distribution for the intact structure
along with all (ten) possible damage scenarios. The ultimate capacity of the intact structure was 11.92 kips and the residual capacity was 5.06 kips . This resulted in a residual strength factor of 0.425 . The governing constraint for the intact ultimate strength was buckling for member 8 . The governing residual capacity was the removal of member 7; when removed the buckling constraint in member 8 was once again reached.

| 10-Bar Truss <br> Ultimate Capacities and Associated Stresses $=\mathrm{A}_{8}=\mathrm{A}_{9}=\mathrm{A}_{10}=1 \mathrm{in}^{3} ; \mathrm{V}_{\text {INTACT }}=1165.69 \mathrm{in}^{4}$ |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Element | Load Level | Stress In Elements (ksi) |  |  |  |  |  |  |  |  |  |
| Removed | (kips) | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 | 10 |
| NONE | 11.92 | -13.72 | 3.68 | 22.04 | 3.68 | 1.88 | 3.68 | 19.41 | -14.31 | 11.66 | -5.20 |
| 1 | 5.06 | --- | 0.96 | 15.18 | 0.96 | 6.02 | 0.96 | 0.00 | -14.31 | 5.80 | -1.35 |
| 2 | 11.44 | -12.76 | --- | 21.56 | 0.00 | -1.32 | 0.00 | 18.05 | -14.31 | 16.18 | 0.00 |
| 3 | 5.89 | -17.68 | 2.95 | --- | 2.95 | -8.84 | 2.95 | 25.00 | 8.33 | 4.17 | -4.17 |
| 4 | 11.44 | -12.76 | 0.00 | 21.56 | --- | -1.32 | 0.00 | 18.05 | -14.31 | 16.18 | 0.00 |
| 5 | 13.22 | -16.32 | 3.10 | 23.34 | 3.10 | --- | 3.10 | 23.08 | -14.31 | 14.31 | -4.39 |
| 6 | 11.44 | -12.76 | 0.00 | 21.56 | 0.00 | -1.32 | --- | 18.05 | -14.31 | 16.18 | 0.00 |
| 7 | 5.06 | 0.00 | 0.96 | 15.18 | 0.96 | 6.02 | 0.96 | --- | -14.31 | 5.80 | -1.35 |
| 8 | 8.84 | -17.68 | 3.50 | 8.84 | 3.50 | -5.33 | 3.50 | 25.00 | --- | 7.54 | -4.96 |
| 9 | 10.12 | -12.46 | 10.12 | 17.90 | 10.12 | 7.78 | 10.12 | 17.62 | -11.00 | --- | -14.31 |
| 10 | 11.44 | -12.76 | 0.00 | 21.56 | 0.00 | -1.32 | 0.00 | 18.05 | -14.31 | 16.18 | --- |

Note: Bolded Stress represent failure of corresponding member via stress limits or buckling limits.
Table 5-Truss damage conditions
An interesting observation happened from the removal of member 5. The ultimate loading for the truss actually increased due to the distribution of forces shedding load from the $8^{\text {th }}$ member. This allowed a higher load to be placed on the structure before the buckling constraint in member 8 was reached.

Next, we will look at the same truss configuration optimized in two different ways. First it was optimized for minimum volume under ultimate capacity requirements and second for minimum volume under both ultimate capacity and residual capacity requirements. The results of these optimizations along with the initial truss results can be found in table 7.

The initial truss has a volume of 1165.69 and an ultimate loading of 11.92 kips due to buckling constraints. The optimized truss for only ultimate loading had a significant decrease in total volume by over half coming in at $560.40 \mathrm{in}^{3}$. This truss configuration sacrifices a significant amount residual capacity from the original intact truss, reducing the residual capacity of the truss by $70.36 \%$

This brings us to the next optimization routine, to consider both ultimate capacity and residual capacity of the system. Optimizing the truss with the same ultimate and residual capacities as the initial structure resulted in a decrease in volume from 1165.69 to 659.73 or a reduction of $43.4 \%$. The residual capacity was then varied while keeping the ultimate capacity requirement fix to show different scenarios of optimization. The truss was able to stay under the initial volume while having a significant increase in residual strength of the system. These gains become capped at a residual factor of around 0.71 due to additional increase of the residual capacity results in an increase of the ultimate capacity. Additional optimizations were run for an increased ultimate capacity and residual capacity.

Due to the low complexity of this problem, both Harmony Search and Gradient methods were able to be used for minimization. As expected the gradient methods provided the global minimum values. The harmony search method is a metaherustic algorithm and does not guarantee a global optimal solution. However, the search routine was

| Optimized for CU |  |  |
| :---: | :---: | :---: |
|  | Gradient <br> Methods | Harmony <br> Search |
| A1 | 0.4109 | 0.4067 |
| A2 | 0.1000 | 0.1000 |
| A3 | 1.0195 | 1.0241 |
| A4 | 0.1000 | 0.1002 |
| A5 | 0.1000 | 0.1058 |
| A6 | 0.1000 | 0.1001 |
| A7 | 0.5811 | 0.5748 |
| A8 | 1.3407 | 1.3522 |
| A9 | 0.6465 | 0.6464 |
| A10 | 0.1000 | 0.1000 |
| Volume | 560.3994 | 561.7659 |

Table 6-Optimization comparison
not far off of the calculus based optimization as seen in table 6. It also implements stochastic optimization methods making the final solution vary slightly from run to run. Due to these factors, a majority of the figures and tables were produced with the calculus methods for more consistent results. The final results for the truss system can be seen in table 7 .

| 10-Bar Truss : Optimization |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Areas (in ${ }^{2}$ ) |  |  |  |  |  |  |  |  |  | Volume $\left(\mathrm{in}^{3}\right)$ | Ultimate Capacity (kips) | Residual <br> Capacity (kips) | Residual Factor |
| $\mathrm{A}_{1}$ | $\mathrm{A}_{2}$ | $\mathrm{A}_{3}$ | A4 | A5 | $\mathrm{A}_{6}$ | $\mathrm{A}_{7}$ | $\mathrm{A}_{8}$ | A9 | $\mathrm{A}_{10}$ | V | Cu | $\mathrm{C}_{\mathrm{R}}$ | R2 |
| Initial |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1.00 | 1165.69 | 11.92 | 5.06 | 0.42 |
| Optimized for CU |  |  |  |  |  |  |  |  |  |  |  |  |  |
| $\begin{gathered} V \rightarrow M \text { Min } \\ C_{U} \geq 11.92 \text { kips } \\ 0.1 \leq \mathrm{A}_{\mathrm{i}} \leq \infty, \mathrm{j}=1,2, \ldots, 10 \end{gathered}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.41 | 0.10 | 1.02 | 0.10 | 0.10 | 0.10 | 0.58 | 1.34 | 0.65 | 0.10 | 560.40 | 11.92 | 1.50 | 0.13 |
| Optimized for CU and CR |  |  |  |  |  |  |  |  |  |  |  |  |  |
| $\begin{gathered} V \rightarrow M i n \\ C_{U} \geq 11.92 \text { kips } \\ C_{R} \geq Q_{R} \\ \leq A_{i} \leq \infty, j=1,2, \ldots, 10 \end{gathered}$ |  |  |  |  |  |  |  |  |  |  |  |  |  |
| 0.61 | 0.20 | 0.95 | 0.20 | 0.34 | 0.20 | 0.86 | 1.18 | 0.57 | 0.29 | 659.73 | 11.92 | 5.06 | 0.42 |
| 0.72 | 0.24 | 0.90 | 0.24 | 0.39 | 0.24 | 1.02 | 1.03 | 0.54 | 0.34 | 688.05 | 11.92 | 6.00 | 0.50 |
| 0.84 | 0.28 | 0.84 | 0.28 | 0.45 | 0.28 | 1.19 | 0.89 | 0.51 | 0.40 | 719.12 | 11.92 | 7.00 | 0.59 |
| 0.96 | 0.32 | 0.96 | 0.32 | 0.51 | 0.32 | 1.36 | 0.91 | 0.49 | 0.45 | 792.33 | 11.92 | 8.00 | 0.67 |
| 1.02 | 0.34 | 1.02 | 0.34 | 0.53 | 0.34 | 1.44 | 0.96 | 0.48 | 0.48 | 835.60 | 11.92 | 8.50 | 0.71 |
| 1.03 | 0.34 | 1.03 | 0.34 | 0.54 | 0.34 | 1.46 | 0.97 | 0.49 | 0.49 | 845.26 | 11.92 | 8.60 | 0.72 |
| 1.20 | 0.40 | 1.20 | 0.40 | 0.63 | 0.40 | 1.70 | 1.13 | 0.62 | 0.57 | 991.25 | 15.00 | 10.00 | 0.67 |
| 1.20 | 0.40 | 1.51 | 0.40 | 0.66 | 0.40 | 1.70 | 1.74 | 0.92 | 0.57 | 1152.42 | 20.00 | 10.00 | 0.50 |

Note : Active constraints are indicated in bold


Table 7 - Truss results

### 6.2 Frame Optimization

The established harmony search optimization algorithm is used to formulate a minimum weight steel frame design. The constraints imposed on the problem will be in accordance to the 2005 AISC-LRFD strength requirements, displacement limitations, and size constraints for beam-column elements. Figure 20 shows the frame configuration, dimensions, loading and grouping of members. The optimum results collected from the harmony search optimization consider the design of rigid and semirigid steel connections and account for linear and nonlinear effects. The obtained results will be compared to an identical structure being optimized with the Genetic Algorithm Technique. The optimization program has discrete variables, which match the AISC shape database. All members with a weight less than 200 lbs were considered in the


Figure 20 - Frame configuration
optimization routine.

### 6.2.2 Frame design parameters

- A36 grade steel
- Young's modulus $\mathrm{E}=30,000 \mathrm{ksi}$
- $\quad$ Allowable total drift $(\mathrm{H} / 300)=1.44 "$
- Allowable interstory drift $(\mathrm{h} / 300)=0.48$ "
- Allowable beam deflection $(\mathrm{L} / 240)=1$ "
- Out of plane effective length for columns $(\mathrm{Ky})=1.0$
- Length of the unbraced compression flange for each column was calculated during the optimization process.
- Floor stringers were assumed to be at $\mathrm{L} / 6$ points of the beam span resulting in the out of plane unbraced length $(\mathrm{Ky})=40^{\prime \prime}$


### 6.2.3 Harmony search design parameters

The following harmony search parameters were selected based on literature recommendations and several trials of the problem to achieve the most refined optimal solution.

- Harmony memory size (HMS) - The value for harmony memory was equal to 25 . This value limits the number of solutions stored in algorithm memory. It was found that a value of 25 had a good tradeoff between run time and accuracy.
- Harmony memory consideration rate (HMCR) - The value for the HMCR was equal to 0.9 , which reflects the probability for selecting a value from memory. Once again, it was found that a $90 \%$ consideration rate provided a good balance between time and accuracy.
- Pitch adjusting rate (PAR) - The pitch-adjusting rate was equal to a value of 0.45 , like the HMCR this value reflects the probability of pitch adjustment. Increased values of PAR caused the solution to converge on non optimal designs where as lower values would not converge on the global maximum.
- Termination criterion - The termination criteria was set as a maximum number of iterations of 8000 . After several trials, it was found that the solution typically converged around 6000. The run time for 8000 iterations with the other parameters listed above takes roughly 75 minutes for the nonlinear analysis. Whereas the linear rigid analysis takes approximately 5 minutes over the same iterations.
- Number of runs - The optimal solution from the harmony search is not a global optimal. Because of this, the optimal solutions will vary from run to run. Ten independent runs were performed to get an average minimum weight structure.


### 6.2.2 Objective function

The objective for the optimization process is to achieve a minimum weight steel frame design. The design variables for this problem are the AISC steel members. The weight of the frame can be expressed in the following equation:

$$
\begin{equation*}
W(x)=\sum_{K=1}^{n g} A_{k} \sum_{\mathrm{i}=1}^{\mathrm{mk}} \rho_{\mathrm{i}} \mathrm{~L}_{\mathrm{i}} \tag{51}
\end{equation*}
$$

where $n g$ is the total number of member groups, mkrepresents the total number of members in that group. The terms $\rho, A_{K}$, and $L_{i}$ represent member density, area, and length respectively.

### 6.2.3 Unconstrained objective penalty function

The unconstrained penalty formula calculates the weight of the new design with an included penalty if any constraints have violation and can be expressed as:

$$
\begin{equation*}
\varphi(x)=W(x)[1+K C]^{\epsilon} \tag{52}
\end{equation*}
$$

where $\mathrm{K}=$ Penalty constant, $\mathrm{C}=$ Constraint violation function and $\in=$ Penalty function exponent. For this design example the values of $\mathrm{K}=1.0$ and $\in=2.0$ were used

### 6.2.4 Constraint violation function formula

$$
\begin{equation*}
C=\sum_{i=1}^{n_{t}} C_{i}^{m d} \sum_{i=1}^{n_{s}} C_{i}^{i d} \sum_{i=1}^{n_{c}} C_{i}^{s c} \sum_{i=1}^{n_{f}} C_{i}^{s b} \sum_{i=1}^{n_{f}} C_{i}^{d} \sum_{i=1}^{n_{c}} C_{i}^{I} \tag{53}
\end{equation*}
$$

where $C_{i}^{m d}$ is the constraint violations for max drift, $C_{i}^{i d}$ is the constraint violations for interstory drift, $C_{i}^{s c}$ is the constraint violations for column sizes, $C_{i}^{s b}$ is the constraint violations for beam sizes, $C_{i}^{d}$ is the constraint violations for deflections and $C_{i}^{I}$ is the
constraint violations for the LRFD interaction equations. The constraint violations are determined based on the following equation:

$$
C_{i}=\left\{\begin{array}{lll}
0 & \text { if } & \beta_{i} \leq 0  \tag{54}\\
\beta_{i} & \text { if } & \beta_{i}>0
\end{array}\right.
$$

### 6.2.5 Drift constraints

$$
\begin{gather*}
\beta_{i}^{t}=\frac{\left|\Delta_{t}\right|}{\left|\Delta_{t}^{u}\right|}-1.0 \leq 0  \tag{55}\\
\beta_{i}^{d}=\frac{\left|\Delta_{i}\right|}{\left|\Delta_{i}^{u}\right|}-1.0 \leq 0 \quad i=1,2, \ldots, n_{s} \tag{56}
\end{gather*}
$$

where $\Delta_{t}$ is the maximum top story displacement, $\Delta_{t}^{u}$ is the allowable top story displacement, $\Delta_{i}$ is interstory displacement, $\Delta_{i}^{u}$ is allowable interstory displacement and $n s$ is the number of stories.

### 6.2.6 Size constraints

$$
\begin{align*}
& \beta_{i}^{s c}=\frac{d_{t}}{d_{b}}-1.0 \leq 0 \quad i=1,2, \ldots, n_{c}  \tag{57}\\
& \beta_{i}^{s b}=\frac{b_{b f}}{b_{c f}}-1.0 \leq 0 \quad i=1,2, \ldots, n_{f} \tag{58}
\end{align*}
$$

where $d_{t}$ is the depth of the top compression member, $d_{b}$ is the depth of the bottom compression member, $b_{b f}$ beam flange width, $b_{c f}$ is the column flange width, $n_{c}$ is the number of compression members and $n_{f}$ is the number of floors.

### 6.2.7 Deflection constraints

$$
\begin{equation*}
\beta_{i}^{d b}=\frac{d_{d b}}{d_{d b}^{u}}-1.0 \leq 0 \quad i=1,2, \ldots, n_{b} \tag{59}
\end{equation*}
$$

where $d_{d b}$ is the deflection of the beam, $d_{d b}^{u}$ is the allowable beam deflection and $n_{b}$ is the number of beams.

### 6.2.8 Strength constraints

Interaction equations from AISC-LRFD were used as the strength constraints for this problem. Doubly and singly symmetric members in flexure and compression should comply with AISC equations H1-1a or $\mathrm{H} 1-1 \mathrm{~b}$.
(a) For $\frac{P_{r}}{P_{c}} \geq 0.2$

$$
\begin{equation*}
\beta_{i}^{I}=\frac{\mathrm{P}_{\mathrm{r}}}{\mathrm{P}_{\mathrm{c}}}+\frac{8}{9}\left(\frac{\mathrm{M}_{\mathrm{rx}}}{\mathrm{M}_{\mathrm{cx}}}+\frac{\mathrm{M}_{\mathrm{ry}}}{\mathrm{M}_{\mathrm{cy}}}\right)-1.0 \leq 0 \tag{60}
\end{equation*}
$$

(b) For $\frac{\mathrm{P}_{\mathrm{r}}}{\mathrm{P}_{\mathrm{c}}}<0.2$

$$
\beta_{i}^{I}=\frac{\mathrm{P}_{\mathrm{r}}}{2 \mathrm{P}_{\mathrm{c}}}+\left(\frac{\mathrm{M}_{\mathrm{rx}}}{\mathrm{M}_{\mathrm{cx}}}+\frac{\mathrm{M}_{\mathrm{ry}}}{\mathrm{M}_{\mathrm{cy}}}\right)-1.0 \leq 0
$$

where $P_{r}$ is the required axial compressive strength, $P_{c}$ is the available axial compressive strength, $M_{r}$ is the required flexural strength, $\mathrm{M}_{\mathrm{c}}$ is the available flexural strength, x and y refer to strong and weak axis bending respectively.

Lateral torsional buckling (LTB) should be checked depending on the unbraced length $L_{b}$ as follows:
(1) $\mathrm{L}_{\mathrm{b}} \leq \mathrm{L}_{\mathrm{p}}$

$$
\begin{equation*}
\mathrm{M}_{\mathrm{n}}=\mathrm{M}_{\mathrm{p}}=\mathrm{F}_{\mathrm{y}} \mathrm{Z}_{\mathrm{x}} \tag{62}
\end{equation*}
$$

(2) $\mathrm{L}_{\mathrm{p}}<L_{\mathrm{b}} \leq \mathrm{L}_{\mathrm{r}}$,

$$
\begin{equation*}
M_{n}=C_{b}\left[M_{p}-\left(M_{p}-0.7 F_{y} S_{x}\right)\left(\frac{L_{b}-L_{p}}{L_{r}-L_{p}}\right)\right] \leq M_{p} \tag{63}
\end{equation*}
$$

(3) $\mathrm{L}_{\mathrm{b}}>\mathrm{L}_{\mathrm{r}}$

$$
\begin{equation*}
\mathrm{M}_{\mathrm{n}}=\mathrm{S}_{\mathrm{x}} \mathrm{~F}_{\mathrm{cr}} \tag{65}
\end{equation*}
$$

where

$$
\begin{gather*}
\mathrm{M}_{\mathrm{r}}=0.7 \mathrm{~F}_{\mathrm{y}} \mathrm{~S}_{\mathrm{x}}  \tag{65}\\
\mathrm{~F}_{\mathrm{cr}}=\frac{\mathrm{C}_{\mathrm{b}} \pi^{2} \mathrm{E}}{\left(\frac{L_{b}}{r_{\mathrm{ts}}}\right)^{2}} \sqrt{1+0.078 \frac{\mathrm{Jc}}{\mathrm{~S}_{\mathrm{x}} \mathrm{~h}_{\mathrm{o}}}\left(\frac{\mathrm{~L}_{\mathrm{b}}}{r_{\mathrm{ts}}}\right)^{2}} \\
\mathrm{~L}_{\mathrm{p}}=1.76 \mathrm{r}_{\mathrm{y}} \sqrt{\frac{\mathrm{E}}{\mathrm{~F}_{\mathrm{y}}}} \\
\mathrm{~L}_{\mathrm{r}}=1.95 \mathrm{r}_{\mathrm{ts}} \frac{\mathrm{E}}{0.7 \mathrm{~F}_{\mathrm{y}}} \sqrt{\frac{\mathrm{Jc}}{\mathrm{~S}_{\mathrm{x}} \mathrm{~h}_{\mathrm{o}}} \sqrt{1+\sqrt{1+6.76\left(\frac{0.7 \mathrm{~F}_{\mathrm{y}}}{\mathrm{E}} \frac{\mathrm{~S}_{\mathrm{x}} h_{o}}{\mathrm{Jc}}\right.}}} \\
\mathrm{r}_{\mathrm{ts}}^{2}=\frac{\sqrt{\mathrm{I}_{\mathrm{y}} \mathrm{C}_{\mathrm{w}}}}{\mathrm{~S}_{\mathrm{x}}} \\
\mathrm{C}_{\mathrm{b}}=\frac{12.5 \mathrm{M}_{\max }}{2.5 \mathrm{M}_{\max }+3 \mathrm{M}_{\mathrm{a}}+4 \mathrm{M}_{\mathrm{b}}+3 \mathrm{M}_{\mathrm{c}}} \leq 3.0
\end{gather*}
$$

where c equals 1 for doubly symmetric shapes and $h_{o}$ is the distance between flange centroids.

### 6.2.8.1 Column strength

AISC Column strength is computed from the following equations:

$$
\mathrm{P}_{\mathrm{n}}=\mathrm{A}_{\mathrm{g}} \mathrm{~F}_{\mathrm{cr}}
$$

(a) $\frac{\mathrm{KL}}{\mathrm{r}} \leq 4.71 \sqrt{\frac{\mathrm{E}}{\mathrm{F}_{\mathrm{y}}}}$

$$
\mathrm{F}_{\mathrm{cr}}=\left[0.658^{\frac{\mathrm{F}_{\mathrm{y}}}{\mathrm{~F}_{\mathrm{e}}}}\right] \mathrm{F}_{\mathrm{y}}
$$

(b) $\frac{\mathrm{KL}}{\mathrm{r}}>4.71 \sqrt{\frac{\mathrm{E}}{\mathrm{F}_{\mathrm{y}}}}$

$$
\begin{gather*}
\mathrm{F}_{\mathrm{cr}}=0.877 \mathrm{~F}_{\mathrm{e}} \\
\mathrm{~F}_{\mathrm{e}}=\frac{\pi^{2} \mathrm{E}}{\left(\frac{\mathrm{KL}}{\mathrm{r}}\right)^{2}} \tag{74}
\end{gather*}
$$

where K is the effective length factor. The effective length factor for unbraced compression flange for each column is calculated throughout the design process from the following equation:

$$
\begin{equation*}
K=\sqrt{\frac{1.6 G_{A} G_{B}+4.0\left(G_{A}+G_{B}\right)+7.50}{G_{A}+G_{B}+7.50}} \tag{75}
\end{equation*}
$$

where $A$ and $B$ represent the top and bottom of the column and the restraint factor $G$ is calculated as

$$
\mathrm{F}_{\mathrm{e}}=\frac{\sum\left(\mathrm{I}_{\mathrm{C}} / \mathrm{L}_{\mathrm{C}}\right)}{\sum\left(\mathrm{I}_{\mathrm{B}} / \mathrm{L}_{\mathrm{B}}\right)}
$$

where $I_{C}$ and $I_{B}$ are moment of inertias and $L_{C}$ and $L_{B}$ are unbraced length of the column and beam respectively.

### 6.2.9 Rigid frame results

Ten separate optimization routines were performed for the structural frame with rigid and semi-rigid connections. From the constraints, it was apparent that the limiting factor for the frame design was strength constraints for rigid connections and a combination of strength and displacement for semi-rigid connections. The lateral drift of the structures were well below acceptable values for all trials with rigid connections. The maximum drift for the rigid design was 0.76 " while the semi-rigid frame was at 1.43 " due to a reduction in the frames stiffness.

The program was set with a max

| Frame Optimization <br> Rigid Connections |  |  |  |
| :---: | :---: | :---: | :---: |
| Frame Analysis | Optimum Weight | Max Improvisation |  |
| 1 | 6908 | 8000 |  |
| 2 | 6461 | 8000 |  |
| 3 | 6973 | 8000 |  |
| 4 | 6791 | 8000 |  |
| 5 | 6754 | 8000 |  |
| 6 | 6729 | 8000 |  |
| 7 | 6804 | 8000 |  |
| 8 | 7065 | 8000 |  |
| 9 | 6539 | 8000 |  |
| 10 | 6430 | 8000 |  |
| 6745 |  |  |  |
| Average (Ibs) | 6430 |  |  |
| Standard Devation (lbs) |  |  |  |
| Minimum (lbs) |  |  |  |

Table 8 - Rigid frame results


Figure 21 - Harmony search iterations
iteration of 8000 . This value has a lot of control over the optimization results as the more iterations the better chance of
a lower weight design. However, it does have a point of diminishing returns. Several extra hours of

| Frame Optimization <br> Semi-Rigid Connections |  |  |  |
| :---: | :---: | :---: | :---: |
| Frame Analysis | Optimum Weight | Max Improvisation |  |
| 1 | 6314 | 8000 |  |
| 2 | 6498 | 8000 |  |
| 3 | 6895 | 8000 |  |
| 4 | 6805 | 8000 |  |
| 5 | 6531 | 8000 |  |
| 6 | 6425 | 8000 |  |
| 7 | 6431 | 8000 |  |
| 8 | 6779 | 8000 |  |
| 9 | 6031 | 8000 |  |
| 10 | 6305 | 8000 |  |
| 6501 |  |  |  |
| Average (lbs) | 605 |  |  |
| Standard Devation (lbs) |  |  |  |
| Minimum (lbs) |  |  |  |

Table 9 - Semi-rigid frame results computer time could be needed for a
few extra thousand iterations that
may only improve the final solution by a fraction of a percent. The design results for the rigid frame and semi-rigid frame analysis are shown in tables 8 and 9 respectively.

The results from table 10 show that harmony search optimization provided a lighter frame compared to the genetic algorithm used by Saka. The results show a reduction in

| Group |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Member | GA (Saka, 2003) |  |  | HS |  |
|  |  | Rigid | Extended end plate | Rigid | Extended end plate |
| 1 | Column | W24X55 | W18X36 | W16X26 | W18X40 |
| 2 | Column | W18X35 | W24X68 | W21X68 | W18X50 |
| 3 | Column | W16X31 | W14X26 | W12X30 | W16X31 |
| 4 | Column | W18X35 | W24X68 | W8X28 | W10X33 |
| 5 | Column | W12X40 | W8X18 | W10X17 | W14X22 |
| 6 | Column | W12X35 | W18X35 | W8X31 | W8X21 |
| 7 | Beam | W16X26 | W16X26 | W16X31 | W12X26 |
| Weight (Ibs) |  | 7404 | 7092 | 6461 | 6031 |

Table 10 - Frame optimization comparison
weight of $14.9 \%$ for the semi-rigid frame and $12.7 \%$ for the rigid frame structure. The results could be improved upon with more efficient harmony search code to allow for a great amount of iterations.

The code was once again implemented for damage tolerance optimization. Due to the extreme number of constraints present in a damage tolerant optimization of the frame structure only one analysis was performed. The same formulation used in the damage tolerant truss example was used for the frame optimization. The initial conditions from the previous frame optimization problem were replicated for the damage tolerant truss. A reduction in member stiffness was used to simulate damage to individual members. In this example members were reduced to $50 \%$ of initial stiffness. The loading for the damage structure was reduced by $50 \%$ to give a residual factor, $\mathrm{R}_{2}$ of 0.50 . The damage tolerant simulation run time took approximately 25 hours. Once again, a more efficient use of code could greatly reduce computational time needed. The damage tolerant frame was found to have an optimal weight of 7059.9 lbs . This represents a weight increase of about $17 \%$ over the non-damage tolerant design.

### 6.3 Damage Tolerance and Reliability

Using the previously stated reliability formulation, the five bar truss shown in figure 22 will be analyzed for reliability. As mentioned earlier, this truss configuration has been optimized for several loading conditions in another report. The member areas used for this problem are highlighted in table 11. The material properties of the structure resemble brittle behavior with a compressive stress limit of -10 ksi and a tensile stress limit of 20 ksi. The modulus of elasticity was deemed deterministic and


Figure 22-5 bar truss

| Optimization Based On Ultimate Capacity and Residual Capacity |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Ultimate Capactiy Cu (ksi) | Residual <br> Capactiy <br> Cr (ksi) | Area (in2) |  |  | Volume <br> (in3) | Residual <br> Factor <br> R2 |
|  |  | A1 | A2 | A3 |  |  |
| 8.256 | 2.477 | 0.951 | 0.248 | 0.99 | 494.925 | 0.300 |
| 8.256 | 3.302 | 0.969 | 0.33 | 0.965 | 499.737 | 0.400 |
| 8.256 | 3.535 | 0.969 | 0.335 | 0.964 | 500.069 | 0.428 |
| 8.256 | 4.128 | 0.946 | 0.413 | 1.168 | 560.609 | 0.500 |
| 8.807 | 4.954 | 0.991 | 0.495 | 1.401 | 643.951 | 0.563 |
| 10.274 | 5.779 | 1.156 | 0.578 | 1.635 | 751.199 | 0.562 |
| 11.743 | 6.605 | 1.321 | 0.661 | 1.868 | 858.583 | 0.562 |

Table 12-5 bar optimization results

| Random <br> Variables | Distribution <br> Type | Mean <br> Value <br> (ksi) | Coefficient <br> of Variation |
| :---: | :---: | :---: | :---: |
| $\sigma_{1}$ | Lognormal | 20 | 0.11 |
| $\sigma_{2}$ | Lognormal | 20 | 0.11 |
| $\sigma_{3}$ | Lognormal | 20 | 0.11 |
| $\sigma_{4}$ | Lognormal | 20 | 0.11 |
| $\sigma_{5}$ | Lognormal | 20 | 0.11 |
| $\mathrm{Q}_{\mathrm{g}}$ | Normal | 1 to 8.256 | 0.2 |
| $\mathrm{Q}_{1}$ | Normal | 1 to 8.256 | 0.2 |

Table 11 - Random variables
this example can be seen in table 12.
To determine the system reliability, there needs to be a mathematical model representing the behavior of the system and the relationship of its components with respect to the overall system. This is accomplished by considering all possible failure modes present to the
 structure. The five bar truss is statically indeterminate to the first degree. This results in

Figure 23 - Series parallel model the configuration needing two members to fail to have a complete structural failure. From this, we can create a series-parallel model for the truss. In this figure, the failure of each individual member is represented by the term $F(i)$. With the intersecting probabilities being represented $\operatorname{by} F(i) \mid F(j)$, which means the failure of member "i" given member " j " has already failed. In total, we should have twenty-five failure paths for this structure. However, when a member is removed, forces throughout the system are redistributed. This can be seen in the removal of member 2, which results in no force present in the fifth member. A zero force member is present once again after the removal of the fifth member. This reduces the total number of failure paths by two with giving a total of twenty-three as shown in figure 23. [2]

The truss reliability was computed using RELSYS (RELiability of SYStems), a
FORTRAN 77 computer program developed by Estes and Frangopol in (1998). The
program works by first computing the reliability of all the system components in a given series-parallel system. The system is then continuously reduced to equivalent components until it is left with one component for the entire system. Series and parallel events are solved separately and equivalent alpha vectors are used to account for the correlation between failure events. [18]

For the truss series parallel system shown, 19 reductions were needed to find the overall system reliability. First, the 18 parallel failure events shown were reduced to a corresponding equivalent event, then the 18 equivalent failure events are represented in a series configuration, which was reduced once again to find the overall failure event. The truss system reliability index and failure probability for several loading magnitudes can be seen in figure 24 and 25 respectively.

As expected probability of the system failing under the 4.128 kip loading was extremely small, so small it can be considered as zero. This trend continued up until a loading of 5


Figure 24 - Reliability index


Figure 25 - Probability of failure
kips. Once the loading surpassed the 5-kip threshold, the probability of failure starts to rise. The ultimate load of the structure at 8.256 kips had a probability of failure right around $50 \%$ and a corresponding reliability index of 0.0.

Correlation between random variables will affect the overall truss reliability. Correlation between the resistances was varied from uncorrelated ( $\rho_{\sigma_{i}, \sigma_{j}}=0.0$ ), $50 \%$ correlated ( $\rho_{\sigma_{i}, \sigma_{j}}=0.5$ ) and fully correlated ( $\rho_{\sigma_{i}, \sigma_{j}}=1.0$ ). Results for each case were plotted and are the results were relatively similar for each correlation case. Full correlation between stresses resulted in the highest reliability index whereas the uncorrelated results gave the lowest reliability index. The effects of correlation between other random variables for the system could also be investigated. This shows the importance of accurately representing the problem data to achieve proper reliability results.

### 6.3.1 Effects of Damage

There are several definitions of structural damage. The term can be defined as any strength deficiency introduced during the design or construction phase of the structure as well as any deterioration of strength caused by external loading and/or environmental conditions during


Figure 26 - Damage effects
the lifetime of the structure. For this example, we will investigate the effects of local damage to specific truss members. The damage has been classified using a damage index associated with the progressive deterioration of the member properties (area). This damage index can range from $0 \leq \delta \leq 1$, with zero representing no damage and one representing complete loss of member. The relationship between cross sectional performance and the damage index relationship can be seen in figure26. For a circular cross-section undergoing uniform damage on the external boundary, the initial area will be reduced by the following equation:

$$
\begin{gathered}
A_{D}=(1-\delta)^{2} * A_{I} \\
\delta=t / r
\end{gathered}
$$

where, $A_{D}$ is the damaged cross-sectional area and $A_{I}$ is the initial area.
Using this representation of damage, each member of the truss was subjected to the full range of damage and the corresponding reliability index was calculated, figure 27. The loading on the truss was considered fixed at 3 kips for the damage conditions. This will ensure that the reliability index of the system will be greater than zero. It can be seen that damage to members 1 and 2 result in


Figure 27 - Damage reliability index
small reduction to the reliability index. Actually, when these members are only slightly damaged the reliability index increases slightly due to the loadings being redistributed to other members. The members that are of interest would be members 3 and 4 since damage to these members cause a significant reduction in the reliability index. It is interesting to note that member 3 should be the first member to fail in the system; however, it does not have the lowest reliability index due to damage. Damage to member 4 actually results in the greatest reduction to the reliability index. This is due to the redistribution of loads when member 4 is removed. The removal of the fourth member puts member 2 and 3 into a significant amount of axial compression resulting in higher failure probabilities for these members. Whereas, the removal of member 3 only puts member 4 into a large amount of axial compress.

### 6.3.2 Measure of Redundancy

Several methods for the quantification of structural redundancy are presented in Frangopol and Curley (1987) and Fu and Frangopol (1990) [2]. The method adapted for this thesis was the probabilistic redundant index approach. This can be expressed by the following equations:

$$
\begin{equation*}
\beta_{R}=\frac{\beta_{\text {intact }}}{\left(\beta_{\text {intact }}-\beta_{\text {damaged }}\right)} \tag{79}
\end{equation*}
$$

where, $\beta_{\text {intact }}$ represents the reliability index of the intact system; and $\beta_{\text {damage }}$ represents the reliability index for the damaged system. The probabilistic redundant index $\beta_{R}$ varies from zero to infinity.


Figure 28 - Redundancy index

With zero indicating a structural collapse and infinity an intact structure. The probabilistic redundancy index for this problem can be seen in figure 28.

An approach to calculating component and system reliability of trusses has been presented. The techniques to quantify and account random variables, redundancy and damage are covered. The optimization methods used in previous work provided good results and correlated with the findings of this thesis. Probabilistic concepts should be utilized when dealing with unknown variables and behavior of the structure needs to be looked at beyond single-element failures by looking at complete structural failure.

## Chapter 7: Conclusion

The goal of this research to develop a computer model capable of optimizing the weight of various steel structures has been presented. The recent developments in metaheuristic optimization algorithms have provided researchers with a wide variety of acceptable methods for optimization. The harmony search algorithm proved to be a well suitable approach for structural optimization. Due to its stochastic random searches, derivative information is unnecessary which allows for the algorithm to easily be implemented. Further research is being done to improve upon this relatively new technique. New approaches have the algorithm constantly changing the search parameters in real time during the optimization process allowing for a more successful code.

The optimum design of steel structures using harmony search algorithm has provided three minimum weight structures. This technique can be very beneficial to both clients and designers from a cost standpoint. The ability to provide a minimum weight deign can be correlated to a reduced cost of the structural system. The designer can also consider damage tolerance in his/her design with minimal change to the original coding. As seen a minimal amount of weight increase could lead to improved structural performance.

Further work could be done to improve the harmony search coding algorithm to increase speed and performance. Also, damage conditions considered in these examples
were simplistic. Therefore, further research could be performed to provide a more accurate representation of damage.

## Chapter 8: Researcher's Biography

R. Bryan Peiffer was born on April 9, 1988 in Harrisburg, Pennsylvania to Scott and Tracy Peiffer. After graduating from Red Land High School, Bryan moved on to attend The Pennsylvania State University to study Architectural Engineering. During his time at Penn State, he became more interested in the studies of building systems, specifically structural systems. He chose to pursue the Structural Option within the Architectural Engineering Program. After graduation from Penn State, Bryan decided to continue his education in the structural field and pursue a Master's of Science in Structural Engineering from Lehigh University.

Bryan currently resides in Hackensack, NJ with his fiancée, Carrie Landis. In early 2014, Bryan ventured out of academia and into the professional world of engineering. He is currently a full time Jr. Engineer at McLaren Engineering. He hopes to continue with his professional development and work towards receiving his Professional Engineering license in the future.

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## Appendix A - W-Shape Selection List

|  |  | W | A | d | $b_{f}$ | $t_{w}$ | $t_{f}$ | $b_{f} / 2 t_{f}$ | $h / t_{w}$ | $I_{\text {x }}$ | $Z_{\text {x }}$ | $S_{x}$ | $r_{x}$ | $I_{y}$ | $z_{y}$ | $S_{y}$ | $r_{y}$ | $J$ | $C_{w}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | W40X199 | 199 | 58.8 | 38.7 | 15.8 | 0.650 | 1.07 | 7.39 | 52.6 | 14900 | 869 | 770 | 16.0 | 695 | 137 | 88.2 | 3.45 | 18.3 | 246000 |
| 2 | W27X194 | 194 | 57.1 | 28.1 | 14.0 | 0.750 | 1.34 | 5.24 | 31.8 | 7860 | 631 | 559 | 11.7 | 619 | 136 | 88.1 | 3.29 | 27.1 | 111000 |
| 3 | W36X194 | 194 | 57.0 | 36.5 | 12.1 | 0.765 | 1.26 | 4.81 | 42.4 | 12100 | 767 | 664 | 14.6 | 375 | 97.7 | 61.9 | 2.56 | 22.2 | 116000 |
| 4 | W14X193 | 193 | 56.8 | 15.5 | 15.7 | 0.890 | 1.44 | 5.45 | 12.8 | 2400 | 355 | 310 | 6.50 | 931 | 180 | 119 | 4.05 | 34.8 | 45900 |
| 5 | W24X192 | 192 | 56.5 | 25.5 | 13.0 | 0.810 | 1.46 | 4.43 | 26.6 | 6260 | 559 | 491 | 10.5 | 530 | 126 | 81.8 | 3.07 | 30.8 | 76300 |
| 6 | W18X192 | 192 | 56.2 | 20.4 | 11.5 | 0.960 | 1.75 | 3.27 | 16.7 | 3870 | 442 | 380 | 8.28 | 440 | 119 | 76.8 | 2.79 | 44.7 | 38000 |
| 7 | W30X191 | 191 | 56.1 | 30.7 | 15.0 | 0.710 | 1.19 | 6.35 | 37.7 | 9200 | 675 | 600 | 12.8 | 673 | 138 | 89.5 | 3.46 | 21.0 | 146000 |
| 8 | W12X190 | 190 | 56.0 | 14.4 | 12.7 | 1.06 | 1.74 | 3.65 | 9.16 | 1890 | 311 | 263 | 5.82 | 589 | 143 | 93.0 | 3.25 | 48.8 | 23600 |
| 9 | W36X182 | 182 | 53.6 | 36.3 | 12.1 | 0.725 | 1.18 | 5.12 | 44.8 | 11300 | 718 | 623 | 14.5 | 347 | 90.7 | 57.6 | 2.55 | 18.5 | 107000 |
| 10 | W21X182 | 182 | 53.6 | 22.7 | 12.5 | 0.830 | 1.48 | 4.22 | 22.6 | 4730 | 476 | 417 | 9.40 | 483 | 119 | 77.2 | 3.00 | 30.7 | 54400 |
| 11 | W40X183 | 183 | 53.3 | 39.0 | 11.8 | 0.650 | 1.20 | 4.92 | 52.6 | 13200 | 774 | 675 | 15.7 | 331 | 88.3 | 56.0 | 2.49 | 19.3 | 118000 |
| 12 | W27X178 | 178 | 52.5 | 27.8 | 14.1 | 0.725 | 1.19 | 5.92 | 32.9 | 7020 | 570 | 505 | 11.6 | 555 | 122 | 78.8 | 3.25 | 20.1 | 98400 |
| 13 | W14X176 | 176 | 51.8 | 15.2 | 15.7 | 0.830 | 1.31 | 5.97 | 13.7 | 2140 | 320 | 281 | 6.43 | 838 | 163 | 107 | 4.02 | 26.5 | 40500 |
| 14 | W24X176 | 176 | 51.7 | 25.2 | 12.9 | 0.750 | 1.34 | 4.81 | 28.7 | 5680 | 511 | 450 | 10.5 | 479 | 115 | 74.3 | 3.04 | 23.9 | 68400 |
| 15 | W18X175 | 175 | 51.4 | 20.0 | 11.4 | 0.890 | 1.59 | 3.58 | 18.0 | 3450 | 398 | 344 | 8.20 | 391 | 106 | 68.8 | 2.76 | 33.8 | 33300 |
| 16 | W30X173 | 173 | 50.9 | 30.4 | 15.0 | 0.655 | 1.07 | 7.04 | 40.8 | 8230 | 607 | 541 | 12.7 | 598 | 123 | 79.8 | 3.42 | 15.6 | 129000 |
| 17 | W36X170 | 170 | 50.0 | 36.2 | 12.0 | 0.680 | 1.10 | 5.47 | 47.7 | 10500 | 668 | 581 | 14.5 | 320 | 83.8 | 53.2 | 2.53 | 15.1 | 98500 |
| 18 | W12X170 | 170 | 50.0 | 14.0 | 12.6 | 0.960 | 1.56 | 4.03 | 10.1 | 1650 | 275 | 235 | 5.74 | 517 | 126 | 82.3 | 3.22 | 35.6 | 20100 |
| 19 | W33X169 | 169 | 49.5 | 33.8 | 11.5 | 0.670 | 1.22 | 4.71 | 44.7 | 9290 | 629 | 549 | 13.7 | 310 | 84.4 | 53.9 | 2.50 | 17.7 | 82400 |
| 20 | W40X167 | 167 | 49.3 | 38.6 | 11.8 | 0.650 | 1.03 | 5.76 | 52.6 | 11600 | 693 | 600 | 15.3 | 283 | 76.0 | 47.9 | 2.40 | 14.0 | 99700 |
| 21 | W21X166 | 166 | 48.8 | 22.5 | 12.4 | 0.750 | 1.36 | 4.57 | 25.0 | 4280 | 432 | 380 | 9.36 | 435 | 108 | 70.0 | 2.99 | 23.6 | 48500 |
| 22 | W24X162 | 162 | 47.8 | 25.0 | 13.0 | 0.705 | 1.22 | 5.31 | 30.6 | 5170 | 468 | 414 | 10.4 | 443 | 105 | 68.4 | 3.05 | 18.5 | 62600 |
| 23 | W27X161 | 161 | 47.6 | 27.6 | 14.0 | 0.660 | 1.08 | 6.49 | 36.1 | 6310 | 515 | 458 | 11.5 | 497 | 109 | 70.9 | 3.23 | 15.1 | 87300 |
| 24 | W36X160 | 160 | 47.0 | 36.0 | 12.0 | 0.650 | 1.02 | 5.88 | 49.9 | 9760 | 624 | 542 | 14.4 | 295 | 77.3 | 49.1 | 2.50 | 12.4 | 90200 |
| 25 | W14X159 | 159 | 46.7 | 15.0 | 15.6 | 0.745 | 1.19 | 6.54 | 15.3 | 1900 | 287 | 254 | 6.38 | 748 | 146 | 96.2 | 4.00 | 19.7 | 35600 |
| 26 | W18X158 | 158 | 46.3 | 19.7 | 11.3 | 0.810 | 1.44 | 3.92 | 19.8 | 3060 | 356 | 310 | 8.12 | 347 | 94.8 | 61.4 | 2.74 | 25.2 | 29000 |
| 27 | W33X152 | 152 | 44.9 | 33.5 | 11.6 | 0.635 | 1.06 | 5.48 | 47.2 | 8160 | 559 | 487 | 13.5 | 273 | 73.9 | 47.2 | 2.47 | 12.4 | 71700 |
| 28 | W12X152 | 152 | 44.7 | 13.7 | 12.5 | 0.870 | 1.40 | 4.46 | 11.2 | 1430 | 243 | 209 | 5.66 | 454 | 111 | 72.8 | 3.19 | 25.8 | 17200 |
| 29 | W36X150 | 150 | 44.3 | 35.9 | 12.0 | 0.625 | 0.940 | 6.37 | 51.9 | 9040 | 581 | 504 | 14.3 | 270 | 70.9 | 45.1 | 2.47 | 10.1 | 82200 |
| 30 | W40X149 | 149 | 43.8 | 38.2 | 11.8 | 0.630 | 0.830 | 7.11 | 54.3 | 9800 | 598 | 513 | 15.0 | 229 | 62.2 | 38.8 | 2.29 | 9.36 | 80000 |
| 31 | W30X148 | 148 | 43.6 | 30.7 | 10.5 | 0.650 | 1.18 | 4.44 | 41.6 | 6680 | 500 | 436 | 12.4 | 227 | 68.0 | 43.3 | 2.28 | 14.5 | 49400 |
| 32 | W27X146 | 146 | 43.2 | 27.4 | 14.0 | 0.605 | 0.975 | 7.16 | 39.4 | 5660 | 464 | 414 | 11.5 | 443 | 97.7 | 63.5 | 3.20 | 11.3 | 77200 |
| 33 | W21X147 | 147 | 43.2 | 22.1 | 12.5 | 0.720 | 1.15 | 5.44 | 26.1 | 3630 | 373 | 329 | 9.17 | 376 | 92.6 | 60.1 | 2.95 | 15.4 | 41100 |
| 34 | W24X146 | 146 | 43.0 | 24.7 | 12.9 | 0.650 | 1.09 | 5.92 | 33.2 | 4580 | 418 | 371 | 10.3 | 391 | 93.2 | 60.5 | 3.01 | 13.4 | 54600 |
| 35 | W14X145 | 145 | 42.7 | 14.8 | 15.5 | 0.680 | 1.09 | 7.11 | 16.8 | 1710 | 260 | 232 | 6.33 | 677 | 133 | 87.3 | 3.98 | 15.2 | 31700 |
| 36 | W18X143 | 143 | 42.0 | 19.5 | 11.2 | 0.730 | 1.32 | 4.25 | 22.0 | 2750 | 322 | 282 | 8.09 | 311 | 85.4 | 55.5 | 2.72 | 19.2 | 25700 |


| 37 | W33X141 | 141 | 41.5 | 33.3 | 11.5 | 0.605 | 0.960 | 6.01 | 49.6 | 7450 | 514 | 448 | 13.4 | 246 | 66.9 | 42.7 | 2.43 | 9.70 | 64400 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 38 | W36X135 | 135 | 39.9 | 35.6 | 12.0 | 0.600 | 0.790 | 7.56 | 54.1 | 7800 | 509 | 439 | 14.0 | 225 | 59.7 | 37.7 | 2.38 | 7.00 | 68100 |
| 39 | W12X136 | 136 | 39.9 | 13.4 | 12.4 | 0.790 | 1.25 | 4.96 | 12.3 | 1240 | 214 | 186 | 5.58 | 398 | 98.0 | 64.2 | 3.16 | 18.5 | 14700 |
| 40 | W30X132 | 132 | 38.8 | 30.3 | 10.5 | 0.615 | 1.00 | 5.27 | 43.9 | 5770 | 437 | 380 | 12.2 | 196 | 58.4 | 37.2 | 2.25 | 9.72 | 42100 |
| 41 | W21X132 | 132 | 38.8 | 21.8 | 12.4 | 0.650 | 1.04 | 6.01 | 28.9 | 3220 | 333 | 295 | 9.12 | 333 | 82.3 | 53.5 | 2.93 | 11.3 | 36000 |
| 42 | W14X132 | 132 | 38.8 | 14.7 | 14.7 | 0.645 | 1.03 | 7.15 | 17.7 | 1530 | 234 | 209 | 6.28 | 548 | 113 | 74.5 | 3.76 | 12.3 | 25500 |
| 43 | W24X131 | 131 | 38.6 | 24.5 | 12.9 | 0.605 | 0.960 | 6.70 | 35.6 | 4020 | 370 | 329 | 10.2 | 340 | 81.5 | 53.0 | 2.97 | 9.50 | 47100 |
| 44 | W33X130 | 130 | 38.3 | 33.1 | 11.5 | 0.580 | 0.855 | 6.73 | 51.7 | 6710 | 467 | 406 | 13.2 | 218 | 59.5 | 37.9 | 2.39 | 7.37 | 56600 |
| 45 | W18X130 | 130 | 38.3 | 19.3 | 11.2 | 0.670 | 1.20 | 4.65 | 23.9 | 2460 | 290 | 256 | 8.03 | 278 | 76.7 | 49.9 | 2.70 | 14.5 | 22700 |
| 46 | W27X129 | 129 | 37.8 | 27.6 | 10.0 | 0.610 | 1.10 | 4.55 | 39.7 | 4760 | 395 | 345 | 11.2 | 184 | 57.6 | 36.8 | 2.21 | 11.1 | 32500 |
| 47 | W30X124 | 124 | 36.5 | 30.2 | 10.5 | 0.585 | 0.930 | 5.65 | 46.2 | 5360 | 408 | 355 | 12.1 | 181 | 54.0 | 34.4 | 2.23 | 7.99 | 38600 |
| 48 | W21X122 | 122 | 35.9 | 21.7 | 12.4 | 0.600 | 0.960 | 6.45 | 31.3 | 2960 | 307 | 273 | 9.09 | 305 | 75.6 | 49.2 | 2.92 | 8.98 | 32700 |
| 49 | W14X120 | 120 | 35.3 | 14.5 | 14.7 | 0.590 | 0.940 | 7.80 | 19.3 | 1380 | 212 | 190 | 6.24 | 495 | 102 | 67.5 | 3.74 | 9.37 | 22700 |
| 50 | W12X120 | 120 | 35.2 | 13.1 | 12.3 | 0.710 | 1.11 | 5.57 | 13.7 | 1070 | 186 | 163 | 5.51 | 345 | 85.4 | 56.0 | 3.13 | 12.9 | 12400 |
| 51 | W18X119 | 119 | 35.1 | 19.0 | 11.3 | 0.655 | 1.06 | 5.31 | 24.5 | 2190 | 262 | 231 | 7.90 | 253 | 69.1 | 44.9 | 2.69 | 10.6 | 20300 |
| 52 | W33X118 | 118 | 34.7 | 32.9 | 11.5 | 0.550 | 0.740 | 7.76 | 54.5 | 5900 | 415 | 359 | 13.0 | 187 | 51.3 | 32.6 | 2.32 | 5.30 | 48300 |
| 53 | W24X117 | 117 | 34.4 | 24.3 | 12.8 | 0.550 | 0.850 | 7.53 | 39.2 | 3540 | 327 | 291 | 10.1 | 297 | 71.4 | 46.5 | 2.94 | 6.72 | 40800 |
| 54 | W30X116 | 116 | 34.2 | 30.0 | 10.5 | 0.565 | 0.850 | 6.17 | 47.8 | 4930 | 378 | 329 | 12.0 | 164 | 49.2 | 31.3 | 2.19 | 6.43 | 34900 |
| 55 | W27X114 | 114 | 33.6 | 27.3 | 10.1 | 0.570 | 0.930 | 5.41 | 42.5 | 4080 | 343 | 299 | 11.0 | 159 | 49.3 | 31.5 | 2.18 | 7.33 | 27600 |
| 56 | W10X112 | 112 | 32.9 | 11.4 | 10.4 | 0.755 | 1.25 | 4.17 | 10.4 | 716 | 147 | 126 | 4.66 | 236 | 69.2 | 45.3 | 2.68 | 15.1 | 6020 |
| 57 | W21X111 | 111 | 32.6 | 21.5 | 12.3 | 0.550 | 0.875 | 7.05 | 34.1 | 2670 | 279 | 249 | 9.05 | 274 | 68.2 | 44.5 | 2.90 | 6.83 | 29200 |
| 58 | W14X109 | 109 | 32.0 | 14.3 | 14.6 | 0.525 | 0.860 | 8.49 | 21.7 | 1240 | 192 | 173 | 6.22 | 447 | 92.7 | 61.2 | 3.73 | 7.12 | 20200 |
| 59 | W30X108 | 108 | 31.7 | 29.8 | 10.5 | 0.545 | 0.760 | 6.89 | 49.6 | 4470 | 346 | 299 | 11.9 | 146 | 43.9 | 27.9 | 2.15 | 4.99 | 30900 |
| 60 | W12X106 | 106 | 31.2 | 12.9 | 12.2 | 0.610 | 0.990 | 6.17 | 15.9 | 933 | 164 | 145 | 5.47 | 301 | 75.1 | 49.3 | 3.11 | 9.13 | 10700 |
| 61 | W18X106 | 106 | 31.1 | 18.7 | 11.2 | 0.590 | 0.940 | 5.96 | 27.2 | 1910 | 230 | 204 | 7.84 | 220 | 60.5 | 39.4 | 2.66 | 7.48 | 17400 |
| 62 | W24X104 | 104 | 30.7 | 24.1 | 12.8 | 0.500 | 0.750 | 8.50 | 43.1 | 3100 | 289 | 258 | 10.1 | 259 | 62.4 | 40.7 | 2.91 | 4.72 | 35200 |
| 63 | W24X103 | 103 | 30.3 | 24.5 | 9.00 | 0.550 | 0.980 | 4.59 | 39.2 | 3000 | 280 | 245 | 10.0 | 119 | 41.5 | 26.5 | 1.99 | 7.07 | 16600 |
| 64 | W27X102 | 102 | 30.0 | 27.1 | 10.0 | 0.515 | 0.830 | 6.03 | 47.1 | 3620 | 305 | 267 | 11.0 | 139 | 43.4 | 27.8 | 2.15 | 5.28 | 24000 |
| 65 | W21X101 | 101 | 29.8 | 21.4 | 12.3 | 0.500 | 0.800 | 7.68 | 37.5 | 2420 | 253 | 227 | 9.02 | 248 | 61.7 | 40.3 | 2.89 | 5.21 | 26200 |
| 66 | W16X100 | 100 | 29.4 | 17.0 | 10.4 | 0.585 | 0.985 | 5.29 | 24.3 | 1490 | 198 | 175 | 7.10 | 186 | 54.9 | 35.7 | 2.51 | 7.73 | 11900 |
| 67 | W10X100 | 100 | 29.3 | 11.1 | 10.3 | 0.680 | 1.12 | 4.62 | 11.6 | 623 | 130 | 112 | 4.60 | 207 | 61.0 | 40.0 | 2.65 | 10.9 | 5150 |
| 68 | W14X99 | 99.0 | 29.1 | 14.2 | 14.6 | 0.485 | 0.780 | 9.34 | 23.5 | 1110 | 173 | 157 | 6.17 | 402 | 83.6 | 55.2 | 3.71 | 5.37 | 18000 |
| 69 | W30X99 | 99.0 | 29.0 | 29.7 | 10.5 | 0.520 | 0.670 | 7.80 | 51.9 | 3990 | 312 | 269 | 11.7 | 128 | 38.6 | 24.5 | 2.10 | 3.77 | 26800 |
| 70 | W18X97 | 97.0 | 28.5 | 18.6 | 11.1 | 0.535 | 0.870 | 6.41 | 30.0 | 1750 | 211 | 188 | 7.82 | 201 | 55.3 | 36.1 | 2.65 | 5.86 | 15800 |
| 71 | W12X96 | 96.0 | 28.2 | 12.7 | 12.2 | 0.550 | 0.900 | 6.76 | 17.7 | 833 | 147 | 131 | 5.44 | 270 | 67.5 | 44.4 | 3.09 | 6.85 | 9410 |
| 72 | W24X94 | 94.0 | 27.7 | 24.3 | 9.07 | 0.515 | 0.875 | 5.18 | 41.9 | 2700 | 254 | 222 | 9.87 | 109 | 37.5 | 24.0 | 1.98 | 5.26 | 15000 |
| 73 | W27X94 | 94.0 | 27.6 | 26.9 | 10.0 | 0.490 | 0.745 | 6.70 | 49.5 | 3270 | 278 | 243 | 10.9 | 124 | 38.8 | 24.8 | 2.12 | 4.03 | 21300 |
| 74 | W21X93 | 93.0 | 27.3 | 21.6 | 8.42 | 0.580 | 0.930 | 4.53 | 32.3 | 2070 | 221 | 192 | 8.70 | 92.9 | 34.7 | 22.1 | 1.84 | 6.03 | 9940 |
| 75 | W14X90 | 90.0 | 26.5 | 14.0 | 14.5 | 0.440 | 0.710 | 10.2 | 25.9 | 999 | 157 | 143 | 6.14 | 362 | 75.6 | 49.9 | 3.70 | 4.06 | 16000 |


| 76 | W30x90 | 90.0 | 26.3 | 29.5 | 10.4 | 0.470 | 0.610 | 8.52 | 57.5 | 3610 | 283 | 245 | 11.7 | 115 | 34.7 | 22.1 | 2.09 | 2.84 | 24000 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 77 | W16X89 | 89.0 | 26.2 | 16.8 | 10.4 | 0.525 | 0.875 | 5.92 | 27.0 | 1300 | 175 | 155 | 7.05 | 163 | 48.1 | 31.4 | 2.49 | 5.45 | 10200 |
| 78 | W10x88 | 88.0 | 26.0 | 10.8 | 10.3 | 0.605 | 0.990 | 5.18 | 13.0 | 534 | 113 | 98.5 | 4.54 | 179 | 53.1 | 34.8 | 2.63 | 7.53 | 4330 |
| 79 | W12X87 | 87.0 | 25.6 | 12.5 | 12.1 | 0.515 | 0.810 | 7.48 | 18.9 | 740 | 132 | 118 | 5.38 | 241 | 60.4 | 39.7 | 3.07 | 5.10 | 8270 |
| 80 | W18X86 | 86.0 | 25.3 | 18.4 | 11.1 | 0.480 | 0.770 | 7.20 | 33.4 | 1530 | 186 | 166 | 7.77 | 175 | 48.4 | 31.6 | 2.63 | 4.10 | 13600 |
| 81 | W27X84 | 84.0 | 24.7 | 26.7 | 10.0 | 0.460 | 0.640 | 7.78 | 52.7 | 2850 | 244 | 213 | 10.7 | 106 | 33.2 | 21.2 | 2.07 | 2.81 | 17900 |
| 82 | W24X84 | 84.0 | 24.7 | 24.1 | 9.02 | 0.470 | 0.770 | 5.86 | 45.9 | 2370 | 224 | 196 | 9.79 | 94.4 | 32.6 | 20.9 | 1.95 | 3.70 | 12800 |
| 83 | W21X83 | 83.0 | 24.4 | 21.4 | 8.36 | 0.515 | 0.835 | 5.00 | 36.4 | 1830 | 196 | 171 | 8.67 | 81.4 | 30.5 | 19.5 | 1.83 | 4.34 | 8630 |
| 84 | W14X82 | 82.0 | 24.0 | 14.3 | 10.1 | 0.510 | 0.855 | 5.92 | 22.4 | 881 | 139 | 123 | 6.05 | 148 | 44.8 | 29.3 | 2.48 | 5.07 | 6710 |
| 85 | W12X79 | 79.0 | 23.2 | 12.4 | 12.1 | 0.470 | 0.735 | 8.22 | 20.7 | 662 | 119 | 107 | 5.34 | 216 | 54.3 | 35.8 | 3.05 | 3.84 | 7330 |
| 86 | W10x77 | 77.0 | 22.7 | 10.6 | 10.2 | 0.530 | 0.870 | 5.86 | 14.8 | 455 | 97.6 | 85.9 | 4.49 | 154 | 45.9 | 30.1 | 2.60 | 5.11 | 3630 |
| 87 | W16X77 | 77.0 | 22.6 | 16.5 | 10.3 | 0.455 | 0.760 | 6.77 | 31.2 | 1110 | 150 | 134 | 7.00 | 138 | 41.1 | 26.9 | 2.47 | 3.57 | 8590 |
| 88 | W24X76 | 76.0 | 22.4 | 23.9 | 8.99 | 0.440 | 0.680 | 6.61 | 49.0 | 2100 | 200 | 176 | 9.69 | 82.5 | 28.6 | 18.4 | 1.92 | 2.68 | 11100 |
| 89 | W18X76 | 76.0 | 22.3 | 18.2 | 11.0 | 0.425 | 0.680 | 8.11 | 37.8 | 1330 | 163 | 146 | 7.73 | 152 | 42.2 | 27.6 | 2.61 | 2.83 | 11700 |
| 90 | W14X74 | 74.0 | 21.8 | 14.2 | 10.1 | 0.450 | 0.785 | 6.41 | 25.4 | 795 | 126 | 112 | 6.04 | 134 | 40.5 | 26.6 | 2.48 | 3.87 | 5990 |
| 91 | W21X73 | 73.0 | 21.5 | 21.2 | 8.30 | 0.455 | 0.740 | 5.60 | 41.2 | 1600 | 172 | 151 | 8.64 | 70.6 | 26.6 | 17.0 | 1.81 | 3.02 | 7410 |
| 92 | W12X72 | 72.0 | 21.1 | 12.3 | 12.0 | 0.430 | 0.670 | 8.99 | 22.6 | 597 | 108 | 97.4 | 5.31 | 195 | 49.2 | 32.4 | 3.04 | 2.93 | 6540 |
| 93 | W18x71 | 71.0 | 20.9 | 18.5 | 7.64 | 0.495 | 0.810 | 4.71 | 32.4 | 1170 | 146 | 127 | 7.50 | 60.3 | 24.7 | 15.8 | 1.70 | 3.49 | 4700 |
| 94 | W24X68 | 68.0 | 20.1 | 23.7 | 8.97 | 0.415 | 0.585 | 7.66 | 52.0 | 1830 | 177 | 154 | 9.55 | 70.4 | 24.5 | 15.7 | 1.87 | 1.87 | 9430 |
| 95 | W21X68 | 68.0 | 20.0 | 21.1 | 8.27 | 0.430 | 0.685 | 6.04 | 43.6 | 1480 | 160 | 140 | 8.60 | 64.7 | 24.4 | 15.7 | 1.80 | 2.45 | 6760 |
| 96 | W14X68 | 68.0 | 20.0 | 14.0 | 10.0 | 0.415 | 0.720 | 6.97 | 27.5 | 722 | 115 | 103 | 6.01 | 121 | 36.9 | 24.2 | 2.46 | 3.01 | 5380 |
| 97 | W10x68 | 68.0 | 19.9 | 10.4 | 10.1 | 0.470 | 0.770 | 6.58 | 16.7 | 394 | 85.3 | 75.7 | 4.44 | 134 | 40.1 | 26.4 | 2.59 | 3.56 | 3100 |
| 98 | W8X67 | 67.0 | 19.7 | 9.00 | 8.28 | 0.570 | 0.935 | 4.43 | 11.1 | 272 | 70.1 | 60.4 | 3.72 | 88.6 | 32.7 | 21.4 | 2.12 | 5.05 | 1440 |
| 99 | W16X67 | 67.0 | 19.6 | 16.3 | 10.2 | 0.395 | 0.665 | 7.70 | 35.9 | 954 | 130 | 117 | 6.96 | 119 | 35.5 | 23.2 | 2.46 | 2.39 | 7300 |
| 100 | W18X65 | 65.0 | 19.1 | 18.4 | 7.59 | 0.450 | 0.750 | 5.06 | 35.7 | 1070 | 133 | 117 | 7.49 | 54.8 | 22.5 | 14.4 | 1.69 | 2.73 | 4240 |
| 101 | W12X65 | 65.0 | 19.1 | 12.1 | 12.0 | 0.390 | 0.605 | 9.92 | 24.9 | 533 | 96.8 | 87.9 | 5.28 | 174 | 44.1 | 29.1 | 3.02 | 2.18 | 5780 |
| 102 | W21X62 | 62.0 | 18.3 | 21.0 | 8.24 | 0.400 | 0.615 | 6.70 | 46.9 | 1330 | 144 | 127 | 8.54 | 57.5 | 21.7 | 14.0 | 1.77 | 1.83 | 5960 |
| 103 | W24X62 | 62.0 | 18.2 | 23.7 | 7.04 | 0.430 | 0.590 | 5.97 | 50.1 | 1550 | 153 | 131 | 9.23 | 34.5 | 15.7 | 9.80 | 1.38 | 1.71 | 4620 |
| 104 | W14X61 | 61.0 | 17.9 | 13.9 | 10.0 | 0.375 | 0.645 | 7.75 | 30.4 | 640 | 102 | 92.1 | 5.98 | 107 | 32.8 | 21.5 | 2.45 | 2.19 | 4710 |
| 105 | W10x60 | 60.0 | 17.7 | 10.2 | 10.1 | 0.420 | 0.680 | 7.41 | 18.7 | 341 | 74.6 | 66.7 | 4.39 | 116 | 35.0 | 23.0 | 2.57 | 2.48 | 2640 |
| 106 | W18×60 | 60.0 | 17.6 | 18.2 | 7.56 | 0.415 | 0.695 | 5.44 | 38.7 | 984 | 123 | 108 | 7.47 | 50.1 | 20.6 | 13.3 | 1.68 | 2.17 | 3850 |
| 107 | W8X58 | 58.0 | 17.1 | 8.75 | 8.22 | 0.510 | 0.810 | 5.07 | 12.4 | 228 | 59.8 | 52.0 | 3.65 | 75.1 | 27.9 | 18.3 | 2.10 | 3.33 | 1180 |
| 108 | W12X58 | 58.0 | 17.0 | 12.2 | 10.0 | 0.360 | 0.640 | 7.82 | 27.0 | 475 | 86.4 | 78.0 | 5.28 | 107 | 32.5 | 21.4 | 2.51 | 2.10 | 3570 |
| 109 | W16X57 | 57.0 | 16.8 | 16.4 | 7.12 | 0.430 | 0.715 | 4.98 | 33.0 | 758 | 105 | 92.2 | 6.72 | 43.1 | 18.9 | 12.1 | 1.60 | 2.22 | 2660 |
| 110 | W21X57 | 57.0 | 16.7 | 21.1 | 6.56 | 0.405 | 0.650 | 5.04 | 46.3 | 1170 | 129 | 111 | 8.36 | 30.6 | 14.8 | 9.35 | 1.35 | 1.77 | 3190 |
| 111 | W24X55 | 55.0 | 16.2 | 23.6 | 7.01 | 0.395 | 0.505 | 6.94 | 54.6 | 1350 | 134 | 114 | 9.11 | 29.1 | 13.3 | 8.30 | 1.34 | 1.18 | 3870 |
| 112 | W21X55 | 55.0 | 16.2 | 20.8 | 8.22 | 0.375 | 0.522 | 7.87 | 50.0 | 1140 | 126 | 110 | 8.40 | 48.4 | 18.4 | 11.8 | 1.73 | 1.24 | 4980 |
| 113 | W18X55 | 55.0 | 16.2 | 18.1 | 7.53 | 0.390 | 0.630 | 5.98 | 41.1 | 890 | 112 | 98.3 | 7.41 | 44.9 | 18.5 | 11.9 | 1.67 | 1.66 | 3430 |
| 114 | W10X54 | 54.0 | 15.8 | 10.1 | 10.0 | 0.370 | 0.615 | 8.15 | 21.2 | 303 | 66.6 | 60.0 | 4.37 | 103 | 31.3 | 20.6 | 2.56 | 1.82 | 2320 |


| 115 | W14X53 | 53.0 | 15.6 | 13.9 | 8.06 | 0.370 | 0.660 | 6.11 | 30.9 | 541 | 87.1 | 77.8 | 5.89 | 57.7 | 22.0 | 14.3 | 1.92 | 1.94 | 2540 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 116 | W12X53 | 53.0 | 15.6 | 12.1 | 10.0 | 0.345 | 0.575 | 8.69 | 28.1 | 425 | 77.9 | 70.6 | 5.23 | 95.8 | 29.1 | 19.2 | 2.48 | 1.58 | 3160 |
| 117 | W21X50 | 50.0 | 14.7 | 20.8 | 6.53 | 0.380 | 0.535 | 6.10 | 49.4 | 984 | 110 | 94.5 | 8.18 | 24.9 | 12.2 | 7.64 | 1.30 | 1.14 | 2570 |
| 118 | W18X50 | 50.0 | 14.7 | 18.0 | 7.50 | 0.355 | 0.570 | 6.57 | 45.2 | 800 | 101 | 88.9 | 7.38 | 40.1 | 16.6 | 10.7 | 1.65 | 1.24 | 3040 |
| 119 | W16X50 | 50.0 | 14.7 | 16.3 | 7.07 | 0.380 | 0.630 | 5.61 | 37.4 | 659 | 92.0 | 81.0 | 6.68 | 37.2 | 16.3 | 10.5 | 1.59 | 1.52 | 2270 |
| 120 | W12X50 | 50.0 | 14.6 | 12.2 | 8.08 | 0.370 | 0.640 | 6.31 | 26.8 | 391 | 71.9 | 64.2 | 5.18 | 56.3 | 21.3 | 13.9 | 1.96 | 1.71 | 1880 |
| 121 | W10X49 | 49.0 | 14.4 | 10.0 | 10.0 | 0.340 | 0.560 | 8.93 | 23.1 | 272 | 60.4 | 54.6 | 4.35 | 93.4 | 28.3 | 18.7 | 2.54 | 1.39 | 2070 |
| 122 | W21X48 | 48.0 | 14.1 | 20.6 | 8.14 | 0.350 | 0.430 | 9.47 | 53.6 | 959 | 107 | 93.0 | 8.24 | 38.7 | 14.9 | 9.52 | 1.66 | 0.803 | 3950 |
| 123 | W14X48 | 48.0 | 14.1 | 13.8 | 8.03 | 0.340 | 0.595 | 6.75 | 33.6 | 484 | 78.4 | 70.2 | 5.85 | 51.4 | 19.6 | 12.8 | 1.91 | 1.45 | 2240 |
| 124 | W8X48 | 48.0 | 14.1 | 8.50 | 8.11 | 0.400 | 0.685 | 5.92 | 15.9 | 184 | 49.0 | 43.2 | 3.61 | 60.9 | 22.9 | 15.0 | 2.08 | 1.96 | 931 |
| 125 | W18X46 | 46.0 | 13.5 | 18.1 | 6.06 | 0.360 | 0.605 | 5.01 | 44.6 | 712 | 90.7 | 78.8 | 7.25 | 22.5 | 11.7 | 7.43 | 1.29 | 1.22 | 1720 |
| 126 | W16X45 | 45.0 | 13.3 | 16.1 | 7.04 | 0.345 | 0.565 | 6.23 | 41.1 | 586 | 82.3 | 72.7 | 6.65 | 32.8 | 14.5 | 9.34 | 1.57 | 1.11 | 1990 |
| 127 | W10X45 | 45.0 | 13.3 | 10.1 | 8.02 | 0.350 | 0.620 | 6.47 | 22.5 | 248 | 54.9 | 49.1 | 4.32 | 53.4 | 20.3 | 13.3 | 2.01 | 1.51 | 1200 |
| 128 | W12X45 | 45.0 | 13.1 | 12.1 | 8.05 | 0.335 | 0.575 | 7.00 | 29.6 | 348 | 64.2 | 57.7 | 5.15 | 50.0 | 19.0 | 12.4 | 1.95 | 1.26 | 1650 |
| 129 | W21X44 | 44.0 | 13.0 | 20.7 | 6.50 | 0.350 | 0.450 | 7.22 | 53.6 | 843 | 95.4 | 81.6 | 8.06 | 20.7 | 10.2 | 6.37 | 1.26 | 0.770 | 2110 |
| 130 | W14X43 | 43.0 | 12.6 | 13.7 | 8.00 | 0.305 | 0.530 | 7.54 | 37.4 | 428 | 69.6 | 62.6 | 5.82 | 45.2 | 17.3 | 11.3 | 1.89 | 1.05 | 1950 |
| 131 | W18X40 | 40.0 | 11.8 | 17.9 | 6.02 | 0.315 | 0.525 | 5.73 | 50.9 | 612 | 78.4 | 68.4 | 7.21 | 19.1 | 10.0 | 6.35 | 1.27 | 0.810 | 1440 |
| 132 | W16X40 | 40.0 | 11.8 | 16.0 | 7.00 | 0.305 | 0.505 | 6.93 | 46.5 | 518 | 73.0 | 64.7 | 6.63 | 28.9 | 12.7 | 8.25 | 1.57 | 0.794 | 1730 |
| 133 | W12X40 | 40.0 | 11.7 | 11.9 | 8.01 | 0.295 | 0.515 | 7.77 | 33.6 | 307 | 57.0 | 51.5 | 5.13 | 44.1 | 16.8 | 11.0 | 1.94 | 0.906 | 1440 |
| 134 | W8X40 | 40.0 | 11.7 | 8.25 | 8.07 | 0.360 | 0.560 | 7.21 | 17.6 | 146 | 39.8 | 35.5 | 3.53 | 49.1 | 18.5 | 12.2 | 2.04 | 1.12 | 726 |
| 135 | W10X39 | 39.0 | 11.5 | 9.92 | 7.99 | 0.315 | 0.530 | 7.53 | 25.0 | 209 | 46.8 | 42.1 | 4.27 | 45.0 | 17.2 | 11.3 | 1.98 | 0.976 | 992 |
| 136 | W14X38 | 38.0 | 11.2 | 14.1 | 6.77 | 0.310 | 0.515 | 6.57 | 39.6 | 385 | 61.5 | 54.6 | 5.87 | 26.7 | 12.1 | 7.88 | 1.55 | 0.798 | 1230 |
| 137 | W16X36 | 36.0 | 10.6 | 15.9 | 6.99 | 0.295 | 0.430 | 8.12 | 48.1 | 448 | 64.0 | 56.5 | 6.51 | 24.5 | 10.8 | 7.00 | 1.52 | 0.545 | 1460 |
| 138 | W18X35 | 35.0 | 10.3 | 17.7 | 6.00 | 0.300 | 0.425 | 7.06 | 53.5 | 510 | 66.5 | 57.6 | 7.04 | 15.3 | 8.06 | 5.12 | 1.22 | 0.506 | 1140 |
| 139 | W12X35 | 35.0 | 10.3 | 12.5 | 6.56 | 0.300 | 0.520 | 6.31 | 36.2 | 285 | 51.2 | 45.6 | 5.25 | 24.5 | 11.5 | 7.47 | 1.54 | 0.741 | 879 |
| 140 | W8X35 | 35.0 | 10.3 | 8.12 | 8.02 | 0.310 | 0.495 | 8.10 | 20.5 | 127 | 34.7 | 31.2 | 3.51 | 42.6 | 16.1 | 10.6 | 2.03 | 0.769 | 619 |
| 141 | W14X34 | 34.0 | 10.0 | 14.0 | 6.75 | 0.285 | 0.455 | 7.41 | 43.1 | 340 | 54.6 | 48.6 | 5.83 | 23.3 | 10.6 | 6.91 | 1.53 | 0.569 | 1070 |
| 142 | W10X33 | 33.0 | 9.71 | 9.73 | 7.96 | 0.290 | 0.435 | 9.15 | 27.1 | 171 | 38.8 | 35.0 | 4.19 | 36.6 | 14.0 | 9.20 | 1.94 | 0.583 | 791 |
| 143 | W16X31 | 31.0 | 9.13 | 15.9 | 5.53 | 0.275 | 0.440 | 6.28 | 51.6 | 375 | 54.0 | 47.2 | 6.41 | 12.4 | 7.03 | 4.49 | 1.17 | 0.461 | 739 |
| 144 | W8X31 | 31.0 | 9.13 | 8.00 | 8.00 | 0.285 | 0.435 | 9.19 | 22.3 | 110 | 30.4 | 27.5 | 3.47 | 37.1 | 14.1 | 9.27 | 2.02 | 0.536 | 530 |
| 145 | W14X30 | 30.0 | 8.85 | 13.8 | 6.73 | 0.270 | 0.385 | 8.74 | 45.4 | 291 | 47.3 | 42.0 | 5.73 | 19.6 | 8.99 | 5.82 | 1.49 | 0.380 | 887 |
| 146 | W10X30 | 30.0 | 8.84 | 10.5 | 5.81 | 0.300 | 0.510 | 5.70 | 29.5 | 170 | 36.6 | 32.4 | 4.38 | 16.7 | 8.84 | 5.75 | 1.37 | 0.622 | 414 |
| 147 | W12X30 | 30.0 | 8.79 | 12.3 | 6.52 | 0.260 | 0.440 | 7.41 | 41.8 | 238 | 43.1 | 38.6 | 5.21 | 20.3 | 9.56 | 6.24 | 1.52 | 0.457 | 720 |
| 148 | W8X28 | 28.0 | 8.25 | 8.06 | 6.54 | 0.285 | 0.465 | 7.03 | 22.3 | 98.0 | 27.2 | 24.3 | 3.45 | 21.7 | 10.1 | 6.63 | 1.62 | 0.537 | 312 |
| 149 | W14X26 | 26.0 | 7.69 | 13.9 | 5.03 | 0.255 | 0.420 | 5.98 | 48.1 | 245 | 40.2 | 35.3 | 5.65 | 8.91 | 5.54 | 3.55 | 1.08 | 0.358 | 405 |
| 150 | W16X26 | 26.0 | 7.68 | 15.7 | 5.50 | 0.250 | 0.345 | 7.97 | 56.8 | 301 | 44.2 | 38.4 | 6.26 | 9.59 | 5.48 | 3.49 | 1.12 | 0.262 | 565 |
| 151 | W12X26 | 26.0 | 7.65 | 12.2 | 6.49 | 0.230 | 0.380 | 8.54 | 47.2 | 204 | 37.2 | 33.4 | 5.17 | 17.3 | 8.17 | 5.34 | 1.51 | 0.300 | 607 |
| 152 | W10X26 | 26.0 | 7.61 | 10.3 | 5.77 | 0.260 | 0.440 | 6.56 | 34.0 | 144 | 31.3 | 27.9 | 4.35 | 14.1 | 7.50 | 4.89 | 1.36 | 0.402 | 345 |
| 153 | W8X24 | 24.0 | 7.08 | 7.93 | 6.50 | 0.245 | 0.400 | 8.12 | 25.9 | 82.7 | 23.1 | 20.9 | 3.42 | 18.3 | 8.57 | 5.63 | 1.61 | 0.346 | 259 |


| 154 | W14X22 | 22.0 | 6.49 | 13.7 | 5.00 | 0.230 | 0.335 | 7.46 | 53.3 | 199 | 33.2 | 29.0 | 5.54 | 7.00 | 4.39 | 2.80 | 1.04 | 0.208 | 314 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 155 | W10X22 | 22.0 | 6.49 | 10.2 | 5.75 | 0.240 | 0.360 | 7.99 | 36.9 | 118 | 26.0 | 23.2 | 4.27 | 11.4 | 6.10 | 3.97 | 1.33 | 0.239 | 275 |
| 156 | W12X22 | 22.0 | 6.48 | 12.3 | 4.03 | 0.260 | 0.425 | 4.74 | 41.8 | 156 | 29.3 | 25.4 | 4.91 | 4.66 | 3.66 | 2.31 | 0.848 | 0.293 | 164 |
| 157 | W8X21 | 21.0 | 6.16 | 8.28 | 5.27 | 0.250 | 0.400 | 6.59 | 27.5 | 75.3 | 20.4 | 18.2 | 3.49 | 9.77 | 5.69 | 3.71 | 1.26 | 0.282 | 152 |
| 158 | W10x19 | 19.0 | 5.62 | 10.2 | 4.02 | 0.250 | 0.395 | 5.09 | 35.4 | 96.3 | 21.6 | 18.8 | 4.14 | 4.29 | 3.35 | 2.14 | 0.874 | 0.233 | 104 |
| 159 | W12X19 | 19.0 | 5.57 | 12.2 | 4.01 | 0.235 | 0.350 | 5.72 | 46.2 | 130 | 24.7 | 21.3 | 4.82 | 3.76 | 2.98 | 1.88 | 0.822 | 0.180 | 131 |
| 160 | W8X18 | 18.0 | 5.26 | 8.14 | 5.25 | 0.230 | 0.330 | 7.95 | 29.9 | 61.9 | 17.0 | 15.2 | 3.43 | 7.97 | 4.66 | 3.04 | 1.23 | 0.172 | 122 |
| 161 | W10x17 | 17.0 | 4.99 | 10.1 | 4.01 | 0.240 | 0.330 | 6.08 | 36.9 | 81.9 | 18.7 | 16.2 | 4.05 | 3.56 | 2.80 | 1.78 | 0.845 | 0.156 | 85.1 |
| 162 | W12X16 | 16.0 | 4.71 | 12.0 | 3.99 | 0.220 | 0.265 | 7.53 | 49.4 | 103 | 20.1 | 17.1 | 4.67 | 2.82 | 2.26 | 1.41 | 0.773 | 0.103 | 96.9 |
| 163 | W8X15 | 15.0 | 4.44 | 8.11 | 4.02 | 0.245 | 0.315 | 6.37 | 28.1 | 48.0 | 13.6 | 11.8 | 3.29 | 3.41 | 2.67 | 1.70 | 0.876 | 0.137 | 51.8 |
| 164 | W10x15 | 15.0 | 4.41 | 9.99 | 4.00 | 0.230 | 0.270 | 7.41 | 38.5 | 68.9 | 16.0 | 13.8 | 3.95 | 2.89 | 2.30 | 1.45 | 0.810 | 0.104 | 68.3 |
| 165 | W12X14 | 14.0 | 4.16 | 11.9 | 3.97 | 0.200 | 0.225 | 8.82 | 54.3 | 88.6 | 17.4 | 14.9 | 4.62 | 2.36 | 1.90 | 1.19 | 0.753 | 0.0704 | 80.4 |
| 166 | W8X13 | 13.0 | 3.84 | 7.99 | 4.00 | 0.230 | 0.255 | 7.84 | 29.9 | 39.6 | 11.4 | 9.91 | 3.21 | 2.73 | 2.15 | 1.37 | 0.843 | 0.0871 | 40.8 |
| 167 | W10×12 | 12.0 | 3.54 | 9.87 | 3.96 | 0.190 | 0.210 | 9.43 | 46.6 | 53.8 | 12.6 | 10.9 | 3.90 | 2.18 | 1.74 | 1.10 | 0.785 | 0.0547 | 50.9 |
| 168 | W8X10 | 10.0 | 2.96 | 7.89 | 3.94 | 0.170 | 0.205 | 9.61 | 40.5 | 30.8 | 8.87 | 7.81 | 3.22 | 2.09 | 1.66 | 1.06 | 0.841 | 0.0426 | 30.9 |

## Appendix B - Harmony Search Sample Frame MATLAB Optimization Code

```
clear all; clc; close all;
tic;
[numerics, strings]=xlsread('Full Catalog Section');
FCS=numerics(:,3:20);
IbeamStr=strings(2:169,1);
NVAR=7; % number of variables
NG=36; % number of ineguality constraints
NH=0; % number of eguality constraints
MaxItr=8000; % maximum number of iterations
HMS=25; % harmony memory size
HMCR=0.9; % harmony consideration rate 0< HMCR <1
PAR=0.45; % minumum pitch adjusting rate
ro=0.28359924220274; %density of steel lb/in3
for kk=1:10
% initialize random HM
for i=1:HMS
    for j=1:NVAR;
        k=randi(length(IbeamStr));
        HM(i,j)=IbeamStr(k);
    end
    for j=1:NVAR
            Z=strcmp(IbeamStr,HM(i,j));
            [r,c]=find(Z);
    end
    for j=1:NVAR;
    Z=strcmp(IbeamStr,HM(i,j));
    [r,c]=find(Z);
    HMnew(i,j)=r;
    end
area=[FCS (HMnew (i, 1), 2) ; FCS(HMnew (i, 2) , 2) ; FCS (HMnew (i, 3) , 2) ; FCS (HMnew (
i,4),2);FCS (HMnew (i,5), 2);FCS(HMnew (i, 6), 2);FCS (HMnew(i, 7), 2)];
depth=[FCS(HMnew (i,1), 3);FCS(HMnew (i,2), 3);FCS (HMnew (i, 3), 3);FCS (HMnew
(i, 4), 3) ; FCS (HMnew (i, 5) , 3) ; FCS (HMnew (i, 6) , 3) ; FCS (HMnew (i, 7) , 3) ];
flange_width=[FCS(HMnew (i,1),4);FCS(HMnew (i, 2), 4);FCS (HMnew (i, 3), 4);FC
S(HMnew (i, 4),4);FCS(HMnew (i,5),4);FCS(HMnew (i, 6),4);FCS(HMnew (i, 7) , 4)]
;
web_thickness=[FCS(HMnew (i, 1), 5); FCS (HMnew (i, 2),5);FCS(HMnew (i, 3), 5); F
CS (HMnew (i, 4),5); FCS (HMnew (i,5) , 5);FCS (HMnew (i, 6) , 5) ; FCS (HMnew (i, 7) , 5)
];
flange_thickness=[FCS(HMnew (i, 1), 6);FCS(HMnew (i, 2), 6);FCS (HMnew (i, 3) ,6
```

```
);FCS(HMnew (i, 4), 6);FCS(HMnew (i, 5) , 6);FCS(HMnew (i, 6), 6);FCS (HMnew (i, 7)
,6)];
inertia=[FCS (HMnew (i, 1), 9); FCS (HMnew (i, 2), 9); FCS (HMnew (i, 3) , 9); FCS (HMn
ew (i, 4), 9); FCS (HMnew (i,5),9);FCS (HMnew (i, 6) , 9); FCS (HMnew (i, 7), 9) ];
    [D,A,Max,Ma,Mb,Mc]=NLstiffness(area,inertia,depth,web_thickness);
[C]=constraint(depth,web_thickness,flange_width,flange_thickness,Max,M
a,Mb,MC,D,A,inertia);
score(i)=fitness(FCS (HMnew (i,1), 2), FCS (HMnew (i, 2), 2),FCS (HMnew (i, 3), 2)
,FCS(HMnew (i,4), 2),FCS (HMnew (i,5), 2),FCS(HMnew (i, 6), 2),FCS (HMnew (i, 7),
2),ro,C);
end
[worstcost worst]=max(score);
HM
score'
% MainHarmony
for t=1:MaxItr;
    %countt=t
        for i=1:NVAR;
            ran1=rand(1);
            if (ran1 < HMCR);
                index = randi(HMS,1);
                NCHV(i) = HM(index,i);
                ran2=rand(1);
                if (ran2 < PAR);
                    NCHV=NCHV;
                    if(ran2 < 0.5);
                        Z=strcmp(IbeamStr,NCHV(i));
                        [r,c]=find(Z);
                            if (r < 168) ;
                            NCHV(i)=IbeamStr(r+1);
                            elseif (r < 167)
                                    NCHV(i)=IbeamStr(r+2);
                            else
                                    NCHV (i)=IbeamStr(r);
                                    end
                else
                        Z=strcmp(IbeamStr,NCHV (i));
                        [r,c]=find(Z);
                            if (r > 1)
                                    NCHV (i)=IbeamStr(r-1);
                                    elseif (r > 2)
                                    NCHV(i)=IbeamStr(r-2);
                            else
                                NCHV (i)=IbeamStr(r);
                        end
                end
```

```
            end
        else
            k=randi(length(IbeamStr));
            NCHV(i)=IbeamStr(k);
        end
    end
for g=1:NVAR
    Z=strcmp(IbeamStr,NCHV (g));
        [r,c]=find(Z);
        NCHVnew (g)=r;
end
area1=[FCS (NCHVnew (1),2);FCS (NCHVnew (2),2);FCS (NCHVnew (3) , 2) ; FCS (NCHVn
ew (4),2);FCS (NCHVnew (5), 2);FCS (NCHVnew (6), 2);FCS (NCHVnew (7) , 2) ];
depth1=[FCS (NCHVnew (1),3);FCS(NCHVnew (2),3);FCS (NCHVnew (3),3);FCS (NCHV
new (4),3);FCS(NCHVnew (5),3);FCS (NCHVnew (6),3);FCS (NCHVnew (7) , 3) ];
flange_width1=[FCS (NCHVnew (1),4);FCS (NCHVnew (2),4);FCS (NCHVnew (3) , 4);F
CS (NCHV̄new (4),4);FCS (NCHVnew (5),4);FCS (NCHVnew (6) , 4);FCS (NCHVnew (7) , 4)
];
web_thicknessl=[FCS(NCHVnew (1),5);FCS(NCHVnew (2),5);FCS (NCHVnew (3) ,5);
FCS (NCHVnew (4),5);FCS (NCHVnew (5) ,5);FCS (NCHVnew (6) ,5);FCS (NCHVnew (7) ,5
)];
flange_thickness1=[FCS(NCHVnew (1) , 6);FCS (NCHVnew (2) , 6);FCS (NCHVnew (3),
6) ; FCS (NCHVnew (4) , 6) ; FCS (NCHVnew (5) , 6) ; FCS (NCHVnew (6) ,6) ; FCS (NCHVnew (7
), 6)];
inertial=[FCS(NCHVnew (1),9);FCS(NCHVnew (2) ,9);FCS (NCHVnew (3) , 9); FCS (NC
HVnew (4) ,9);FCS (NCHVnew (5),9);FCS (NCHVnew (6) , 9) ; FCS (NCHVnew (7) , 9) ];
[D1,A1,Max1,Ma1,Mb1,Mc1]=NLstiffness(area1,inertia1,depth1,web_thickne
ss1);
[C1]=constraint(depth1,web_thickness1,flange_width1,flange_thickness1,
Max1,Ma1,Mb1,Mc1,D1,A1,iner}tia1)
NEWfit=fitness(FCS (NCHVnew (1), 2),FCS (NCHVnew (2),2),FCS (NCHVnew (3) , 2), F
CS (NCHVnew (4) , 2),FCS (NCHVnew (5), 2),FCS (NCHVnew (6) , 2),FCS (NCHVnew (7) , 2)
,ro,C1);
    if NEWfit < worstcost
        HM (worst,:)=NCHV;
        score(worst)=NEWfit;
    end
    [worstcost worst]=max(score);
        [a b]=min(score);
        xmin=HM(b,:);
```

```
    fmin=score(b);
    cc(t)=fmin;
end
[x y]=min(score);
HMmin=HM(y,:)
for g=1:NVAR
    Z=strcmp(IbeamStr,HMmin(g));
    [r,c]=find(Z);
    HMminNUM(g)=r;
end
area2=[FCS(HMminNUM(1),2);FCS(HMminNUM(2),2);FCS(HMminNUM(3),2);FCS(HM
minNUM(4),2);FCS(HMminNUM(5),2);FCS(HMminNUM(6),2);FCS (HMminNUM(7) ,2)]
;
depth2=[FCS(HMminNUM (1),3);FCS(HMminNUM(2),3);FCS(HMminNUM (3),3);FCS (H
MminNUM(4),3);FCS(HMminNUM(5), 3);FCS(HMminNUM(6),3);FCS(HMminNUM(7),3)
];
flange width2=[FCS(HMminNUM(1),4);FCS (HMminNUM(2),4);FCS(HMminNUM (3),4
);FCS(HMminNUM(4),4);FCS(HMminNUM(5),4);FCS(HMminNUM(6),4);FCS(HMminNU
M(7),4)];
web thickness2=[FCS(HMminNUM(1),5);FCS (HMminNUM (2) , 5) ; FCS (HMminNUM(3),
5);\overline{FCS (HMminNUM (4),5);FCS(HMminNUM (5),5);FCS(HMminNUM(6),5);FCS(HMminN}
UM(7),5)];
flange_thickness2=[FCS(HMminNUM(1),6);FCS(HMminNUM(2),6);FCS (HMminNUM(
3) , 6);FCS (HMminNUM (4),6);FCS (HMminNUM (5) , 6);FCS (HMminNUM (6) , 6);FCS (HMm
inNUM(7),6)];
inertia2=[FCS(HMminNUM(1),9);FCS(HMminNUM(2),9);FCS (HMminNUM(3),9);FCS
(HMminNUM (4),9);FCS(HMminNUM (5),9);FCS (HMminNUM (6) ,9);FCS (HMminNUM(7),
9)];
[D2,A2,Max2,Ma2,Mb2,Mc2]=NLstiffness(area2,inertia2,depth2,web_thickne
ss2);
[C2]=constraint(depth2,web_thickness2,flange_width2,flange_thickness2,
Max2,Ma2,Mb2,Mc2,D2,A2,inertia2);
    SCOREmin=score(y)
    D2 (160)
    plot(cc);
    toc
end
```


## Appendix C - Nonlinear Stiffness MATLAB Code

```
function [Displacementi,Forces,TotalForces,StiffnessMatrix] =
StiffnessNONlin(Displacement,pe,area,inertia,depth,web)
```

```
nel=60; %number of elmenets
```

nel=60; %number of elmenets
nnel=2; %number of nodes per element
nnel=2; %number of nodes per element
ndof=3; %number of DOFs per node
ndof=3; %number of DOFs per node
edof=nnel*ndof; %number of DOFs per element
edof=nnel*ndof; %number of DOFs per element
nnode=57; %total number of nodes in system
nnode=57; %total number of nodes in system
sdof=nnode*ndof;

```

```

area(5) area(5) area(6) area(5) area(7) area(7) area(7) area(7)
area(7) area(7) area(7) area(7)];
I=[inertia(1) inertia(2) inertia(1) inertia(1) inertia(2) inertia(1)
inertia(1) inertia(2) inertia(1) inertia(1) inertia(2) inertia(1)
inertia(7) inertia(7) inertia(7) inertia(7) inertia(7) inertia(7)
inertia(7) inertia(7) inertia(3) inertia(4) inertia(3) inertia(3)
inertia(4) inertia(3) inertia(3) inertia(4) inertia(3) inertia(3)
inertia(4) inertia(3) inertia(7) inertia(7) inertia(7) inertia(7)
inertia(7) inertia(7) inertia(7) inertia(7) inertia(5) inertia(6)
inertia(5) inertia(5) inertia(6) inertia(5) inertia(5) inertia(6)
inertia(5) inertia(5) inertia(6) inertia(5) inertia(7) inertia(7)
inertia(7) inertia(7) inertia(7) inertia(7) inertia(7) inertia(7)];
d=[depth(1) depth(2) depth(1) depth(1) depth(2) depth(1) depth(1)
depth(2) depth(1) depth(1) depth(2) depth(1) depth(7) depth(7)
depth(7) depth(7) depth(7) depth(7) depth(7) depth(7) depth(3)
depth(4) depth(3) depth(3) depth(4) depth(3) depth(3) depth(4)
depth(3) depth(3) depth(4) depth(3) depth(7) depth(7) depth(7)
depth(7) depth(7) depth(7) depth(7) depth(7) depth(5) depth(6)
depth(5) depth(5) depth(6) depth(5) depth(5) depth(6) depth(5)
depth(5) depth(6) depth(5) depth(7) depth(7) depth(7) depth(7)
depth(7) depth(7) depth(7) depth(7)];
M=[pe(3,13) pe(6,16) pe(3,17) pe(6,20) pe(3,33) pe(6,36) pe(3,37)
pe(6,40) pe(3,53) pe(6,56) pe(3,57) pe(6,60)];
N=[pe(4,:)];

```
```

C1=1.83*10^-3;
C2=1.04*10^-4;
C3=6.38*10^-6;
tp=0.685;
dg=[depth(7)+6];
tf=1;
Kcon1=(dg^-2.4)*(tp^-0.4)*(tf^-1.5);
Kcon2=(dg^-2.4)* (tp^-0.4)* (tf^-1.5);
Kcon3=(dg^-2.4)*(tp^-0.4)*(tf^-1.5);
Kcon4=(dg^-2.4)*(tp^-0.4)*(tf^-1.5);
Kcon5=(dg^-2.4)*(tp^-0.4)* (tf^-1.5);
Kcon6=(dg^-2.4)*(tp^-0.4)*(tf^-1.5);
Theta1=C1* (Kcon1*M(1))^1+C2*(Kcon1*M(1))^3+C3* (Kcon1*M(1))^5;
Theta2=C1* (Kcon2*M(2))^1+C2*(Kcon2*M(2))^3+C3* (Kcon2*M(2))^5;
Theta3=C1*(Kcon2*M(3))^1+C2* (Kcon2*M(3))^3+C3* (Kcon2*M(3))^5;
Theta4=C1* (Kcon1*M(4))^1+C2*(Kcon1*M(4))^3+C3* (Kcon1*M(4) )^5;
Theta 5=C1* (Kcon3*M(5))^1+C2*(Kcon3*M(5))^3+C3* (Kcon3*M(5))^5;
Theta6=C1*(Kcon 4*M(6))^1+C2* (Kcon 4*M(6))^3+C3* (Kcon 4*M(6))^5;
Theta7=C1* (Kcon4*M(7) )^1+C2* (Kcon4*M(7))^3+C3* (Kcon4*M(7) )^5;
Theta8=C1*(Kcon3*M(8))^1+C2* (Kcon3*M(8))^3+C3* (Kcon3*M(8))^5;
Theta9=C1*(Kcon5*M(9))^1+C2*(Kcon5*M(9))^3+C3*(Kcon5*M(9))^5;
Theta10=C1*(Kcon6*M(10))^1+C2* (Kcon6*M(10))^3+C3* (Kcon6*M(10))^5;
Theta11=C1* (Kcon6*M(11))^1+C2* (Kcon6*M(11))^3+C3* (Kcon6*M(11))^5;
Theta12=C1* (Kcon5*M(12))^1+C2* (Kcon5*M(12))^3+C3* (Kcon5*M(12))^5;
R1=M(1)/Thetal;

```
```

R2=M(2)/Theta2;
R3=M(3)/Theta3;
R4=M(4)/Theta4;
R5=M(5)/Theta5;
R6=M(6)/Theta6;
R7=M(7)/Theta7;
R8=M(8)/Theta8;
R9=M(9)/Theta9;
R10=M(10)/Theta10;
R11=M(11)/Theta11;
R12=M(12)/Theta12;
r=ones (60,2);
r(13,1)=1/(1+((3*E*I(7))/(R1*60)));
r(16,2)=1/(1+((3*E*I(7))/(R2*60)));
r(17,1)=1/(1+((3*E*I (7))/(R3*60)));
r(20,2)=1/(1+((3*E*I (7))/(R4*60)));
r(33,1)=1/(1+((3*E*I (7))/(R5*60)));
r(36,2)=1/(1+((3*E*I(7))/(R6*60)));
r(37,1)=1/(1+((3*E*I (7))/(R7*60)));
r(40,2)=1/(1+((3*E*I(7))/(R8*60)));
r(53,1)=1/(1+((3*E*I (7))/(R9*60)));
r(56,2)=1/(1+((3*E*I (7))/(R10*60)));
r(57,1)=1/(1+((3*E*I (7))/(R11*60)));
r(60,2)=1/(1+((3*E*I (7))/(R12*60)));
L=zeros(60,1);
% Member 1
i=1;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end

```
```

K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 2
i=2;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LOcM(:,:,i)'*k(:,:,i)*LOcM(:,:,i);
% Member 3
i=3;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:, i)'*k(:,:,i)*LocM(:,:, i);
% Member 4

```
```

    i=4;r1(i)=r(i,1);
    r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 5
i=5;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 6
i=6;
r1(i)=r(i,1);
r2(i)=r(i,2);

```
```

N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 7
i=7;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 8
i=8;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1) )^2 + (MC(i,4)-MC(i,2))^2);

```
```

te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i, 4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);
% Member 9
i=9;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 10
i=10;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));

```
```

cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 11
i=11;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 12
i=12;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));

```
```

cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 13
i=13;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }1
i=14;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1) )^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);

```
```

k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);
% Member 15
i=15;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 16
i=16;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);

```
```

nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LOcM(:,:,i);
% Member 17
i=17;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 18
i=18;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';

```
```

LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 19
i=19;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
Cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);
% Member 20
i=20;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;

```
```

        for g = 1:6;
            if index(g) == j;
                LocM(g,j,i) = 1;
            end
        end
    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }2
i=21;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 22
i=22;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;

```
```

                        LocM(g,j,i) = 1;
            end
        end
    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 23
i=23;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 24
i=24;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end

```
```

        end
    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);
% Member 25
i=25;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 26
i=26;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end

```
```

K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 27
i=27;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LOcM(:,:,i)'*k(:,:,i)*LOcM(:,:,i);
% Member 28
i=28;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LOcM(:,:,i)'*k(:,:,i)*LOCM(:,:,i);
% Member 29

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```

    i=29;
    r1(i)=r(i,1);
    r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 30
i=30;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 31
i=31;
r1(i)=r(i,1);

```
```

r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i) *cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 32
i=32;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 33
i=33;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);

```
```

L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i) *cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 34
i=34;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }3
i=35;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));

```
```

si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 36
i=36;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LOcM(:,:,i);
% Member 37
i=37;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));

```
```

gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i) *cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 38
i=38;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 39
i=39;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));

```
```

ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*\operatorname{LocM}(:,:,i);
% Member 40
i=40;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 41
i=41;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);

```
```

nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 42
i=42;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LOCM(:,:,i);
% Member 43
i=43;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);

```
```

index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 44
i=44;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:, i);
% Member 45
i=45;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));

```
```

    for j = 1:3*nnode;
    for g = 1:6;
            if index(g) == j;
                LocM(g,j,i) = 1;
            end
        end
    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);
% Member 46
i=46;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i) +gi(:,:,i) *cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 47
i=47;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^^2 +(MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;

```
```

                if index(g) == j;
                    LocM(g,j,i) = 1;
                end
        end
    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 48
i=48;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 49
i=49;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;

```
```

                end
        end
    end
    K(:, :, i) =LOcM(:, :, i)'*k(:, :, i)*LOCM(:, :, i);
% Member 50
i=50;
rl(i)=r(i,1);
r2(i)=r(i, 2);
N(i)=pe(4,i);
L(i) = sqrt((MC (i, 3) -MC(i,1) )^2 + (MC(i,4)-MC(i, 2) )^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:, :,i)= Cei(r1(i),r2(i),L(i));
gi(:, :,i) = Gi(N(i),L(i));
Cgi(:, :,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:, :,i)*Cei(:, :,i) +gi(:, :,i)*Cgi(:, :,i);
k(:, :,i)=te(:, :,i)'*ki(:, :, i) *te(:, :, i);
nd(1)=nodes(i,1);
nd(2)=nodes(i, 2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i)=zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LOCM(g,j,i)=1;
end
end
end
K(:, :, i) =LOCM(:, :, i)'*k(:, :, i)*LOCM(:, :, i);
% Member 51
i=51;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC (i, 3)-MC(i,1) )^2 +(MC(i,4)-MC(i, 2) )^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
Cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:, :,i) = Gi(N(i),L(i));
Cgi(:, :,i)= Cgi(r1(i),r2(i),L(i));
ki(:, :,i)=si(:, :,i)*Cei(:, :,i) +gi(:, :,i)*Cgi(:, :,i);
k(:, :,i)=te(:, :, i)'*ki(:, :, i)*te(:, :, i);
nd(1)=nodes(i,1);
nd(2)=nodes(i, 2);
index=feeldof(nd, nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LOCM(g,j,i)=1;
end
end

```
```

    end
    K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }5
i=52;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*Cei(:,:,i) +gi(:,:,i)*Cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 53
i=53;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);

```
```

% Member 54
i=54;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 55
i=55;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 + (MC (i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }5
i=56;

```
```

    r1(i)=r(i,1);
    r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member }5
i=57;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 58
i=58;
r1(i)=r(i,1);
r2(i)=r(i,2);

```
```

N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd (2)=nodes (i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 59
i=59;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1))^2 +(MC(i,4)-MC(i,2))^2);
te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:,:,i);
% Member 60
i=60;
r1(i)=r(i,1);
r2(i)=r(i,2);
N(i)=pe(4,i);
L(i) = sqrt((MC(i,3)-MC(i,1) )^2 + (MC(i,4)-MC(i,2))^2);

```
```

te(:,:,i) = Te(MC(i,1),MC(i,2),MC(i,3),MC(i,4));
si(:,:,i) = Si(E,A(i),L(i),I(i));
cei(:,:,i)= Cei(r1(i),r2(i),L(i));
gi(:,:,i) = Gi(N(i),L(i));
Cgi(:,:,i)= Cgi(r1(i),r2(i),L(i));
ki(:,:,i)=si(:,:,i)*cei(:,:,i)+gi(:,:,i)*cgi(:,:,i);
k(:,:,i)=te(:,:,i)'*ki(:,:,i)*te(:,:,i);
nd(1)=nodes(i,1);
nd(2)=nodes(i,2);
index=feeldof(nd,nnel,ndof).';
LocM(:,:,i) = zeros(6,(3*nnode));
for j = 1:3*nnode;
for g = 1:6;
if index(g) == j;
LocM(g,j,i) = 1;
end
end
end
K(:,:,i)=LocM(:,:,i)'*k(:,:,i)*LocM(:, :,i);

```
StiffnessMatrix=K (:,:,1) +K (: , : , 2) +K (: , : , 3) +K (:, : , 4) +K (: , : , 5) +K (: , : , 6) +
\(\mathrm{K}(:,:, 7)+\mathrm{K}(:,:, 8)+\mathrm{K}(:,:, 9)+\mathrm{K}(:,: 10)+\mathrm{K}(:,:, 11)+\mathrm{K}(:,:, 12)+\mathrm{K}(:,:, 13)+\mathrm{K}(:\)
\(,:, 14)+K(:,:, 15)+K(:,:, 16)+K(:,: 17)+K(:,:, 18)+K(:,: 19)+K(:,:, 20)+K(:\)
\(,:, 21)+K(:,:, 22)+K(:,:, 23)+K(:,:, 24)+K(:,:, 25)+K(:,: 26)+K(:,:, 27)+K(:\)
\(,: 28)+\mathrm{K}(:,:, 29)+\mathrm{K}(:,:, 30)+\mathrm{K}(:,:, 31)+\mathrm{K}(:,:, 32)+\mathrm{K}(:,:, 33)+\mathrm{K}(:,:, 34)+\mathrm{K}(:\)
\(,:, 35)+K(:,:, 36)+K(:,:, 37)+K(:,:, 38)+K(:,:, 39)+K(:,: 40)+K(:,: 41)+K(:\)
\(,:, 42)+K(:,:, 43)+K(:,:, 44)+K(:,:, 45)+K(:,:, 46)+K(:,: 47)+K(:,:, 48)+K(:\)
\(,:, 49)+K(:,:, 50)+K(:,:, 51)+K(:,:, 52)+K(:,:, 53)+K(:,:, 54)+K(:,:, 55)+K(:\)
,: , 56) \(+\mathrm{K}(:,:, 57)+\mathrm{K}(:,:, 58)+\mathrm{K}(:,:, 59)+\mathrm{K}(:,:, 60)\);
Kcc=StiffnessMatrix(163:171,163:171);
Kcu=StiffnessMatrix(163:171,1:162);
Kuc=StiffnessMatrix(1:162,163:171);
Kuu=StiffnessMatrix(1:162,1:162);
ff=(zeros(sdof,1));
\(\mathrm{ff}(28)=(1 / 10) * 8\);
\(\mathrm{ff}(82)=(1 / 10) * 8\);
\(\mathrm{ff}(136)=(1 / 10) * 4\);
Pfef=zeros (6,60);
M11 \(=(((1 / 10) * 0.22 * 60 \wedge 2) / 12) *(3 * r(13,1) *(2-r(13,2)) /(4-\)
\(r(13,1) * r(13,2)))\);
M12 \(=(((1 / 10) * 0.22 * 60 \wedge 2) / 12) *(3 * r(13,2) *(2-r(13,1)) /(4-\)
\(r(13,1) * r(13,2)))\);
\(\mathrm{M} 21=(((1 / 10) * 0.22 * 60 \wedge 2) / 12) *(3 * r(16,1) *(2-r(16,2)) /(4-\)
\(r(16,1) * r(16,2)))\);
M22 \(=(((1 / 10) * 0.22 * 60 \wedge 2) / 12) *(3 * r(16,2) *(2-r(16,1)) /(4-\)
\(r(16,1) * r(16,2)))\);
M31 \(=(((1 / 10) * 0.22 * 60 \wedge 2) / 12) *(3 * r(17,1) *(2-r(17,2)) /(4-\)
\(r(17,1) * r(17,2)))\);
```

M32 =(((1/10)*0.22*60^2)/12)*(3*r(17, 2)*(2-r(17,1))/(4-
r(17,1)*r(17,2)));
M41=(((1/10)*0.22*60^2)/12)* (3*r (20,1)* (2-r (20, 2)) / (4-
r(20,1)*r(20,2)));
M42=(((1/10)*0.22*60^2)/12)* (3*r (20,2)* (2-r (20,1)) / (4-
r(20,1)*r(20,2)));
M51=(((1/10)*0.22*60^2) /12)* (3*r(33,1)* (2-r(33,2)) / (4-
r(33,1)*r(33,2)));
M52=(((1/10)*0.22*60^2) /12)* (3*r (33,2)* (2-r (33,1)) / (4-
r(33,1)*r(33,2)));
M61=(((1/10)*0.22*60^2) /12)* (3*r(36,1)* (2-r (36,2)) / (4-
r(36,1)*r(36,2)));
M62=(((1/10)*0.22*60^2) /12)* (3*r(36,2)*(2-r(36,1)) / (4-
r(36,1)*r(36,2)));
M71=(((1/10)*0.22*60^2)/12)* (3*r(37,1)* (2-r (37,2))/(4-
r(37,1)*r(37,2)));
M72=(((1/10)*0.22*60^2) /12)*(3*r(37,2)* (2-r (37,1))/(4-
r(37,1)*r(37,2)));
M81=(((1/10)*0.22*60^2) /12)* (3*r(40,1)*(2-r(40,2)) /(4-
r(40,1)*r(40,2)));
M82=(((1/10)*0.22*60^2) /12)* (3*r(40, 2)* (2-r(40,1))/(4-
r(40,1)*r(40,2)));
M91=(((1/10)*0.17*60^2) /12)*(3*r(53,1)* (2-r(53,2)) / (4-
r(53,1)*r(53,2)));
M92=(((1/10)*0.17*60^2) /12)* (3*r (53,2)* (2-r (53,1)) / (4-
r(53,1)*r(53,2)));
M101=(((1/10)*0.17*60^2)/12)*(3*r(56,1)*(2-r (56,2))/(4-
r(56,1)*r(56,2)));
M102=(((1/10)*0.17*60^2)/12)*(3*r(56,2)*(2-r(56,1))/(4-
r(56,1)*r(56,2)));
M111=(((1/10)*0.17*60^2)/12)*(3*r (57,1)*(2-r(57,2))/(4-
r(57,1)*r(57,2)));
M112=(((1/10)*0.17*60^2)/12)* (3*r(57,2)* (2-r(57,1))/(4-
r(57,1)*r(57,2)));
M121=(((1/10)*0.17*60^2) /12)*(3*r(60,1)*(2-r(60,2))/(4-
r(60,1)*r(60,2)));
M122=(((1/10)*0.17*60^2) /12)*(3*r (60,2)*(2-r(60,1))/(4-
r(60,1)*r(60,2)));

```
```

V11=(((1/10)*0.22*60)/2)+(M11+M12)/60;

```
V11=(((1/10)*0.22*60)/2)+(M11+M12)/60;
V12=(((1/10)*0.22*60)/2)+-(M11+M12)/60;
V12=(((1/10)*0.22*60)/2)+-(M11+M12)/60;
V21=(((1/10)*0.22*60)/2)+(M21+M22)/60;
V21=(((1/10)*0.22*60)/2)+(M21+M22)/60;
V22 = (((1/10)*0.22*60)/2)+-(M21+M22)/60;
V22 = (((1/10)*0.22*60)/2)+-(M21+M22)/60;
V31=(((1/10)*0.22*60)/2)+(M31+M32)/60;
V31=(((1/10)*0.22*60)/2)+(M31+M32)/60;
V32=(((1/10)*0.22*60)/2)+-(M31+M32)/60;
V32=(((1/10)*0.22*60)/2)+-(M31+M32)/60;
V41=(((1/10)*0.22*60)/2)+(M41+M42)/60;
V41=(((1/10)*0.22*60)/2)+(M41+M42)/60;
V42=(((1/10)*0.22*60)/2)+-(M41+M42)/60;
V42=(((1/10)*0.22*60)/2)+-(M41+M42)/60;
V51=(((1/10)*0.22*60)/2)+(M51+M52)/60;
V51=(((1/10)*0.22*60)/2)+(M51+M52)/60;
V52=(((1/10)*0.22*60)/2)+-(M51+M52)/60;
V52=(((1/10)*0.22*60)/2)+-(M51+M52)/60;
V61=(((1/10)*0.22*60)/2)+(M61+M62)/60;
V61=(((1/10)*0.22*60)/2)+(M61+M62)/60;
V62=(((1/10)*0.22*60)/2)+-(M61+M62)/60;
V62=(((1/10)*0.22*60)/2)+-(M61+M62)/60;
V71=(((1/10)*0.22*60)/2)+(M71+M72)/60;
```

V71=(((1/10)*0.22*60)/2)+(M71+M72)/60;

```
```

V72=(((1/10)*0.22*60)/2)+-(M71+M72)/60;
V81=(((1/10)*0.22*60)/2)+(M81+M82)/60;
V82 = (((1/10)*0.22*60)/2) +-(M81+M82)/60;
V91=(((1/10)*0.17*60)/2)+(M91+M92)/60;
V92=(((1/10)*0.17*60)/2)+-(M91+M92)/60;
V101=(((1/10)*0.17*60)/2)+(M101+M102)/60;
V102=(((1/10)*0.17*60)/2)+-(M101+M102)/60;
V111=(((1/10)*0.17*60)/2)+(M111+M112)/60;
V112=(((1/10)*0.17*60)/2)+-(M111+M112)/60;
V121=(((1/10)*0.17*60)/2)+(M121+M122)/60;
V122=(((1/10)*0.17*60)/2)+-(M121+M122)/60;
Mw22=(((1/10)*0.22*60^2)/12);
Mw17=(((1/10)*0.17*60^2)/12);
Vw22=(((1/10)*0.22*60)/2);
Vw17=(((1/10)*0.17*60)/2);
w_22=[0;Vw22;Mw22;0;Vw22;-Mw22];
w_17=[0;Vw17;Mw17;0;Vw17;-Mw17];
NF1=[0;V11;M11;0;V12;-M12];
NF2=[0;V21;M21;0;V22;-M22];
NF3=[0;V31;M31;0;V32;-M32];
NF4=[0;V41;M41;0;V42;-M42];
NF5=[0;V51;M51;0;V52;-M52];
NF6=[0;V61;M61;0;V62;-M62];
NF7=[0;V71;M71;0;V72;-M72];
NF8=[0;V81;M81;0;V82;-M82];
NF9=[0;V91;M91;0;V92;-M92];
NF10=[0;V101;M101;0;V102;-M102];
NF11=[0;V111;M111;0;V112;-M112];
NF12=[0;V121;M121;0;V122;-M122];
Pfef(:,13)=NF1;
Pfef(:,14)=w_22;
Pfef(:,15)=w_22;
Pfef(:,16)=N\overline{F}2;
Pfef(:,17)=NF3;
Pfef(:,18)=w_22;
Pfef(:,19)=w_22;
Pfef(:,20)=NF4;
Pfef(:,33)=NF5;
Pfef(:,34)=w 22;
Pfef(:,35)=w_22;
Pfef(:,36)=NF6;
Pfef(:,37)=NF7;
Pfef(:,38)=w_22;
Pfef(:,39) =w-22;
Pfef(:,40)=N\overline{F}8;
Pfef(:,53)=NF9;
Pfef(:,54)=w_17;

```
```

Pfef(:,55)=w_17;
Pfef(:,56)=NF10;
Pfef(:,57)=NF11;
Pfef(:,58)=w_17;
Pfef(:,59)=w_17;
Pfef(:,60)=NF12;
for i = 1:60
pf=transpose(te(:,:,i))*Pfef(:,i);
QQ=transpose(LocM(:,:,i))*pf;
pfi(:,i)=QQ;
end
Pf=sum(pfi,2);
Pu=ff-Pf;
Uc=ff(163:171);
Uu=Kuu^-1*(Pu(1:162)-Kuc*Uc);
D1=(zeros(171,1));
D1 (163:171,1)=Uc;
D1(1:162,1)=Uu;
Displace=D1;
Reactions=Kcu*Uu+Kcc*Uc;
R=Reactions;
for j = 1:60
u=LocM(:,:,j)*D1;
ue=te(:,:,j)*u;
p=(ki(:,:,j)*ue)+Pfef(:,j);
p(6) =p (6)*-1;
pei(:,j)=p;
end
Displacementi=Displace+Displacement;
Forces=pei;
TotalForces=pei+pe;
end

```
```

