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# Elastic-plastic analysis of plane flexural frames

H. B. Harrison

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LEHIGH UNIVERSITY

DEPARTMENT OF CIVIL ENGINEERING

ELASTIC - PLASTIC PLANE FRAME ANALYSIS

- (1) A compiled Fortran programme is now available which will carry out a first-order elastic-plastic analysis of plane frames using the GE225 computer. The method was outlined in a paper by C. K. Wang of Wisconsin in the December, 1963 Journal of the Structural Division, ASCE. The analysis is by the Displacement method with a sequential determination of the location and load factor when plastic hinges are formed. At each stage, the deformations and bending moments are printed and the angular rotations of plastic hinges are output after the collapse mechanism has been found. The maximum size frame that can be analysed using the GE225 would consist of 10 members with 15 degrees of freedom. Point application of loads, moments only can be considered and load application positions must be treated as joints. For larger problems the source programme is available in Fortran so that other larger computers could be used. In its present form, the programme does not take into account directly the effects of axial load upon stiffness or plastic moment, and only flexural members can be accommodated, so that braced frames cannot be analysed.

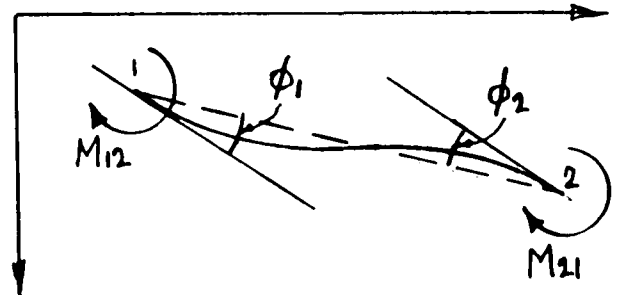
- (2) In using the compiled programme on the GE225 machine, all that is necessary is to prepare the relevant data in the appropriate form as below:
- (a) Card 1. Frame number (for identification) in Fortran Format I5.
  - (b) Card 2. Degrees of freedom and TWICE the number of members in Format 2I5.
  - (c) Cards 3,---J The statics matrix (all elements) in Format 7F10.4.

- (d) Cards L,---N      The stiffness matrix in Format F10.4 in the following sequence: S11, S12, S22, S23, S33, S34,-----Snn.
- (e) Cards M----P      Elements of the plastic moment vector in Format 7F10.4.
- (f) Card Q              The unit load set number in Format I5 (must be positive).
- (g) Cards R----S      The unit load set in Format 7F10.4  
For more load sets, continue with set number, the stiffness matrix again, and the load set
- (h) Card T              If no more load sets, a negative integer in Format I5.
- (i) Cards W----W      Further frames, repeating the sequence (a) to (h).
- (j) Card X              For no further frames, a negative integer in Format I5. Hence the run will end when two negative integers have been read sequentially.
- (k) Cards Y,Z          Blank cards.

Note on Fortran Format    examples of I5: Card 1 bbbb2, Card 2 bbb35, Card 3 bbb-4  
                                   (b denotes space)    examples of 2I5: Card 1 bbbb2bbb35, Card 2 bbb23bbbb3  
   examples of F10.4: Card 1 bbb-63.832, Card 2 bbb0.0bbbb  
   for 7F10.4, 7 such entries per data card are permitted,  
   each within    field of width 10.

(3) The Stiffness Matrix    This matrix represents the assembled load-displacement relationship for all the frame elements and the form chosen follows the usual slope-deflection convention. For a member 1-2,

$$\begin{bmatrix} M_{12} \\ M_{21} \end{bmatrix} = \begin{bmatrix} \frac{4EI}{L} & \frac{2EI}{L} \\ \frac{2EI}{L} & \frac{4EI}{L} \end{bmatrix} \times \begin{bmatrix} \phi_1 \\ \phi_2 \end{bmatrix}$$



For the whole frame,  $M = S \cdot \emptyset$  where  $M$  and  $\emptyset$  are column vectors and  $S$  is a triple-diagonal square matrix of order equal to twice the number of frame members. The ends of each member must be identified by numbers quite distinct from any joint numbering system.

The Statics Matrix This matrix represents the equilibrium equations for the frame.

$$W = A \cdot M$$

$W$  is a column vector with as many elements as degrees of freedom and  $M$  is the column vector as above, listing the internal moments at member ends. Hence there will be twice as many elements in  $M$  as there are frame members.

The Plastic Moment Matrix is a column vector similar to  $M$  listing the moment capacities available at each end of all members.

The unit load set is the matrix  $W$  as above.

Note: The statics matrix  $A$  is the transpose of the more familiar displacement matrix  $A^T$  which relates kinematically the relative to absolute deformations.

$$\emptyset = A^T \cdot \Delta$$

Both matrices are readily assembled and if this is done, a useful check on mistakes is available.

Limitations:

In addition to the general limitation on frame size that can be accommodated by the Lehigh computer, there are two other important limitations

on the efficiency of the programme.

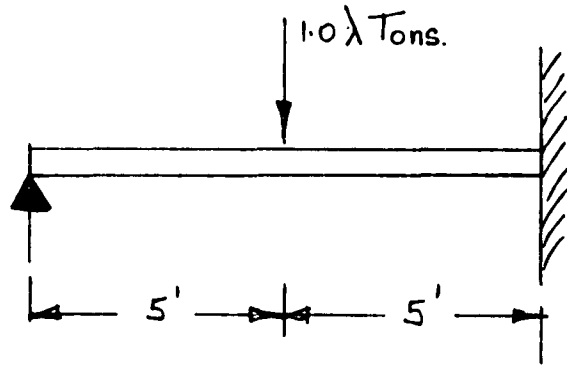
- (1) It is assumed that a plastic hinge once formed stays formed and if this is not the case for a frame, the results will not be of much use except that the load factor at collapse will err on the safe side. This follows from the fact that equilibrium and yield conditions will have been satisfied but not so the mechanism condition.
- (2) It is also assumed that no strain reversal takes place in the frames of progressively deteriorated stiffness that are analysed. However, the printed output is sufficient to indicate whether this phenomenon has occurred. It is not likely that this weakness will seriously limit the usefulness of the programme as the phenomenon has occurred only once in ten frames that have been analyzed by the author.

H. B. Harrison  
March 26, 1965

Example: Frame No. 2

Degrees of freedom 3

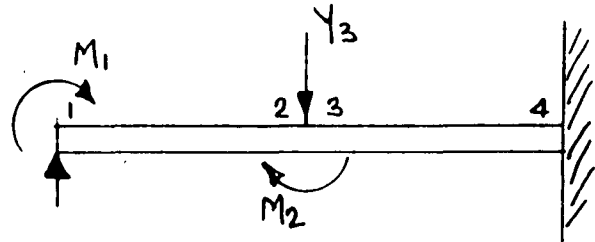
Number of members 2



Assume  $M_p = 40 \text{ Ton ft.}$

Assume  $EI = 5000 \text{ Ton-ft.}^2$

Number member ends as shown



Statics Matrix (external loads as functions of stress resultants).

$$\begin{bmatrix} M_1 \\ M_2 \\ Y_3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 1 & 0 \\ -0.2 & -0.2 & 0.2 & 0.2 \end{bmatrix} \times \begin{bmatrix} M_{12} \\ M_{21} \\ M_{34} \\ M_{43} \end{bmatrix}$$

Stiffness Matrix (stress resultants as functions of relative end slopes).

$$\begin{bmatrix} M_{12} \\ M_{21} \\ M_{34} \\ M_{43} \end{bmatrix} = \begin{bmatrix} 4000 & 2000 & 0 & 0 \\ 2000 & 4000 & 0 & 0 \\ 0 & 0 & 4000 & 2000 \\ 0 & 0 & 2000 & 4000 \end{bmatrix} \times \begin{bmatrix} \phi_{12} \\ \phi_{21} \\ \phi_{34} \\ \phi_{43} \end{bmatrix}$$

The computer output for this example follows.

ELASTIC PLASTIC FIRST ORDER ANALYSIS OF FRAME NO. 2

THE STATICS MATRIX A

ROW 1	0.1000000E 01	0.	0.	0.
ROW 2	0.	0.1000000E 01	0.1000000E 01	0.
ROW 3	-0.2000000E-00	-0.2000000E-00	0.2000000E-00	0.2000000E-00

THE STIFFNESS MATRIX S

ROW 1	0.4000000E 04	0.2000000E 04	0.	0.
ROW 2	0.2000000E 04	0.4000000E 04	0.	0.
ROW 3	0.	0.	0.4000000E 04	0.2000000E 04
ROW 4	0.	0.	0.2000000E 04	0.4000000E 04

THE EXTERNAL LOAD VECTOR SET NO. 1

ROW 1 0.  
ROW 2 0.  
ROW 3 0.1000000E 01

DEFLECTIONS DUE TO UNIT LOADS

ROW 1 0.6250000E-03  
ROW 2 -0.1562500E-03  
ROW 3 0.1822917E-02

MOMENTS DUE TO UNIT LOADS

ROW 1 0.  
ROW 2 -0.1562500E 01  
ROW 3 0.1562500E 01  
ROW 4 0.1875000E 01

PLASTIC HINGE NO. 1 FORMED AT POINT 4

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE( 1)	0.2133333E 02	0.2133333E 02	
DEFLECTION	ADDITIONAL	CUMULATIVE	
X( 1)	0.1333333E-01	0.1333333E-01	
X( 2)	-0.3333333E-02	-0.3333333E-02	
X( 3)	0.3888889E-01	0.3888889E-01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
M( 1)	0.	0.	40.0000
M( 2)	-33.3333	-33.3333	40.0000
M( 3)	33.3333	33.3333	40.0000
M( 4)	40.0000	40.0000	40.0000

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.1250000E-02
ROW 2	0.
ROW 3	0.4166667E-02

MOMENTS DUE TO UNIT LOADS

ROW 1	0.
ROW 2	-0.2500000E 01
ROW 3	0.2500000E 01
ROW 4	0.

PLASTIC HINGE NO. 2 FORMED AT POINT 2

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE( 2)	0.2666667E 01	0.2400000E 02	
DEFLECTION	ADDITIONAL	CUMULATIVE	
X( 1)	0.3333333E-02	0.1666667E-01	
X( 2)	0.	-0.3333333E-02	
X( 3)	0.1111111E-01	0.5000000E-01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
M( 1)	0.	0.	40.0000
M( 2)	-6.6667	-40.0000	40.0000
M( 3)	6.6667	40.0000	40.0000
M( 4)	0.	40.0000	40.0000

DIVISION BY ZERO IN INVERSION

COLLAPSE MECHANISM HAS BEEN REACHED

HINGE ROTATIONS	
AT POINT( 1)	-0.2182787E-10
AT POINT( 2)	-0.2910383E-10
AT POINT( 3)	-0.4365575E-10
AT POINT( 4)	-0.3333333E-02



Plastic Design in High Strength Steel

THE ELASTIC-PLASTIC ANALYSIS  
OF PLANE FLEXURAL FRAMES

by

H. B. Harrison

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## SYNOPSIS

An account is given of a Fortran program for the elastic-plastic analysis of plane flexural frames. The program has been developed from one first written by Professor C. K. Wang and has proved to be useful in the study of steel structures.

With a minimum of input data, the program will enable a computer to carry out a series of elastic analyses of a steel structure. The position of each plastic hinge will be determined as it is formed and the load factor and deformed state of the structure will be output as each such event occurs. When the collapse mechanism is found, the rotations at each plastic hinge are computed as well as the deformations and load factor at the outset of failure.

Frames of moderate size can be analyzed by currently operating machines but an upper limit will exist for the frame size that can be handled by any given machine.

The limitations of the program are discussed in detail and several examples are given of its application.

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## I INTRODUCTION

Professor C. K. Wang of the University of Wisconsin first described the basic principles of a computer program to analyze elastic-plastic structures in 1963<sup>(1)</sup> and made available to the author the Fortran coding of his scheme which was in a form suited to the CDC 1604 machine of Wisconsin. In modifying the program to suit the GE 225 machine at Lehigh University it soon became apparent that with the reduced storage capacity available, some attention should be paid to the reduction of the dimensioned arrays used by the program so that frames of reasonable size could be accommodated. The efficiency of the program has been improved in various ways which will be outlined in this report but the basic operating principles and solution techniques used originally by Wang are retained and due acknowledgement is paid for the ingenious way in which he has achieved his goal.

Basically, the program will carry out a series of first-order elastic analyses of a frame in which free hinges are successively introduced at those sections at which localized plastic hinges are assumed to develop at the load system is increased uniformly. Accordingly, it can be appreciated that the program must incorporate two distinct capabilities. The first is a system of "bookkeeping" in which a record is kept of the moments existing at all possible plastic hinge positions in a frame. The moments are compared with the available plastic moment capacity to detect whether or not the next plastic hinge is to form at any given position. The second capability is the utilization of a form of first order elastic analysis which can be applied simply and successively to frames of deteriorated stiffness as hinges are inserted. The type of displacement analysis

described by Clough<sup>(2)</sup> and used by Wang is well suited for this purpose. Brief explanations will be given of both sections of the program since the functioning and limitations of the scheme can only be understood in their light.

## II ELASTIC-PLASTIC ANALYSIS

It is often the case that the form of an analysis carried out by hand would not be a desirable one to program for a computer. Neal and Symonds<sup>(3)</sup> have proposed a method for estimating the deformations near collapse of rigid frames and it has been used by Heyman<sup>(4)</sup> and Vickery<sup>(5)</sup> in a study of the effects of deformation and strain hardening on the collapse load. Heyman<sup>(6)</sup> has subsequently used a different approach based on Virtual Work to achieve the same end. In all methods, the mechanism of failure is found previously and the deformations at failure are determined by first finding the position where the last-to-form plastic hinge would occur. These methods have had the common aim of avoiding the onerous computation of load factors and deformations as each plastic hinge is formed when the load intensity is progressively increased. This latter approach is probably the best to use with a computer as intuitive judgements are eliminated. In such a method of computation, once it has been decided that a plastic hinge exists at some position, the next stage in the analysis concerns the same initial frame with a free hinge at the position nominated, but subjected to a new loading system. The new system would consist of the original set of unit loads together with a moment of the full plastic value acting as an external action on the ends of the members meeting at the "hinge". The method is demonstrated in Fig. 1 for a propped cantilever where the results of the first elastic analysis shown in Fig. 1 (b) indicate that the first plastic hinge will form at position C. Inserting a free hinge at C, it can be seen that the second and final analysis shown in Fig. 1 (c) is that of a simply supported beam with an extra external action, namely the moment  $M_p$ , acting at C. This approach presents no problems for

a hand solution, but it would be inefficient for a machine solution because of the necessity of providing for the extra degree of freedom and the corresponding new loading term in the dimensioning of the various matrices affected by the degree of freedom. If provision had to be made for an extra degree of freedom at every position where a hinge was likely to form, a small frame would rapidly fill the available data storage capacity of a computer.

The alternative system used by Wang does not involve the same difficulties and is illustrated for the propped cantilever shown again in Fig. 2. The results of the first elastic analysis are shown in Fig. 2(b) and in row 4 of Table I. The load factors in row 5, obtained by dividing the available moment capacity at each position by the unit moment at the same position, determine where the first plastic hinge will form. This will be the case at that position where the load factor is smallest as shown in row 6. The moments at all positions when the first hinge has formed are shown in row 8 and the residual moment capacity is shown in row 9 of the table and also in Fig. 2(c). With a free hinge inserted at position 4 in Fig. 2, the frame is again analyzed for the original loading system as in row 10 with the load factors determined by dividing the residual moment capacities by the unit load moments. It is in this respect, illustrated in Fig. 2(c) that the machine solution devised by Wang differs from the hand solution technique.

It can also be seen from Table I that the procedure is essentially cyclical. It is feasible to calculate the deformation at each stage but these results have not been included in the tabulation. A collapse mechanism will have been reached in the analysis when the structure has

been converted into a mechanism. The numerical indication of such a phenomenon can be in several forms. It may be that the coefficients in the stiffness equations would form a singular matrix so that zero division would be encountered in an attempted solution and would end the analysis. If this does not occur, the computed deformations would be very large which would indicate that the load-deflection diagram has become horizontal. Wang has explained the computer indications of frame failure in reference (1) though some of his collapse criteria have been eliminated in the present program for reasons which will be explained later.

The method chosen for systematizing an elastic-plastic analysis has been explained and its success as part of a computer program will obviously depend upon the provision of a method of elastic analysis which will deal in a simple fashion with the insertion of hinges in rigid frames.



Table I

Numerical Analysis of Propped Cantilever Problem

Position in Beam (Fig. 2)	1.	2.	3.	4.
1. Initial moment capacity	40	40	40	40
2. Cumulative moments to date	0	0	0	0
3. Available moment capacity	40	40	40	40
4. <u>Elastic Analysis 1</u>				
Moments due to unit loading	0	-1.56	1.56	1.83
5. Load factors (row 3 ÷ row 4)*	—	25.64	25.64	21.33
6. Smallest load factor (SLG)				21.33
7. Unit moments x SLG	0	-33.33	33.33	40
8. Cumulative moments to date	0	-33.33	33.33	40
9. Avail. capacity (row 1 - row 8)	40	6.67	6.67	0
10. <u>Elastic Analysis 2</u>				
Moments due to unit loading with hinge at position 4	0	-2.5	2.5	0
11. Load factors (row 9 ÷ row 10)*	—	2.66	2.66	*
12. Smallest load factor (SLG)		<u>2.66</u>		
13. Unit moments x SLG	0	-6.67	6.67	0
14. Cumulative moments to date	0	-40	40	40
15. Avail. capacity (row 1 - row 14)	40	0	0	0
16. <u>Elastic Analysis 3</u> with hinges at positions 4 and 2	(Either zero division or very large deformations will result)			

\* Positions where moment is near zero are not included in search for smallest load factor to avoid premature zero division stop.

(The computer output for this problem is shown in Appendix B)

### III ELASTIC FLEXURAL ANALYSIS

The displacement method of frame analysis can be formulated in many forms, all with the common characteristic that the load-displacement behavior of a frame as a whole is built up from a knowledge of the load-displacement relationship for its component members. In the case of a flexural frame, the elementary component will be a straight prismatic member as shown at (a) in Fig. 3 and if axial and shear stiffnesses are assumed infinite, the load displacement relationships take the form of the simple slope-deflection equations,

$$M_{AB} = \frac{2EI}{L}(2\phi_{AB} + \phi_{BA})$$

$$M_{BA} = \frac{2EI}{L}(\phi_{AB} + 2\phi_{BA})$$

which can be expressed in the matrix form,

$$\begin{bmatrix} M_{AB} \\ M_{BA} \end{bmatrix} = \begin{bmatrix} \frac{4EI}{L} & \frac{2EI}{L} \\ \frac{2EI}{L} & \frac{4EI}{L} \end{bmatrix} \cdot \begin{bmatrix} \phi_{AB} \\ \phi_{BA} \end{bmatrix}$$

or

$$(SR_{AB}) = (S_{AB}) \cdot (\phi_{AB}) \quad (1)$$

For all the members of a frame, the similar equations for each member may be assembled in the matrix equation,

$$\begin{bmatrix} (SR_{AB}) \\ (SR_{BC}) \\ (SR_{CD}) \end{bmatrix} = \begin{bmatrix} (S_{AB}) & \cdot & \cdot \\ \cdot & (S_{BC}) & \cdot \\ \cdot & \cdot & (S_{CD}) \end{bmatrix} \cdot \begin{bmatrix} (\phi_{AB}) \\ (\phi_{BC}) \\ (\phi_{CD}) \end{bmatrix}$$

or

$$(SR) = (S) \cdot (\phi) \quad (2)$$

where (S) is called the member stiffness matrix and (SR) will be a column matrix or vector listing the moments acting at the ends of all frame members. It is usually a simple matter to write down the equations of statics which relate these moments (called stress resultants) to the applied loads.

$$(W) = (A) \cdot (SR) \quad (3)$$

The load vector (W) must have as many terms as the degree of freedom of the structure since deformations are measured by the movement of loads (whether real or virtual) in a displacement analysis. If the degree of freedom is  $\bar{L}$  and the number of members is NM, then the statics matrix (A) will be of order  $\bar{L} \times 2NM$ . Only for a statically determinate structure will  $\bar{L} = 2NM$  so that inversion of (A) is then possible and the stress resultants will be known in terms of loads without any further analysis. Finally, the relative deformations within each member ( $\phi$ ) can be expressed in terms of movements of the loads (X) by a kinematics matrix C,

$$(\phi) = (C) \cdot (X) \quad (4)$$

and it can be shown<sup>(1)</sup> that the matrix (C) is the transpose of the statics matrix (A). Hence, the load-displacement equations for the whole structure can be expressed,

$$(W) = (A) \cdot (S) \cdot (A^T) \cdot (X) \quad (5)$$

where the triple matrix product  $(A \cdot S \cdot A^T)$  is the stiffness matrix (K) of the frame. For a given set of loads (W), the displacements can be determined by standard equation solution programs. Thereafter, the moments at the ends of each frame member can be computed from Eqs. (2) and (4),

$$(SR) = (S) \cdot (A^T) \cdot (X) \quad (6)$$

This form of first order frame analysis can accommodate the modification associated with the insertion of a hinge within a structure. There are two ways in which the modification can be made. The obvious way is to consider the extra degree of freedom involved and to add a row to the matrix (A) (and a corresponding column to  $A^T$ ) leaving the member stiffness matrix (S) unchanged. It has been explained earlier that this approach would be impracticable in a computer program as all possible changes in the degree of freedom would have to be accounted for in the initial dimensioning and establishment of the statics matrix (A). The alternative approach adopted by Wang was to keep (A) and effectively  $\bar{L}$  unchanged and modify the member stiffness matrix (S). The procedure can be understood by referring to Fig. 3(b). If a hinge is present at the end A of member AB, the slope-deflection equations become,

$$M_{AB} = 0 = \frac{2EI}{L} (2\phi_{AB} + \phi_{BA}) ; M_{BA} = \frac{2EI}{L} (\phi_{AB} + \phi_{BA}) \text{ and hence,}$$

$$\begin{bmatrix} M_{AB} \\ M_{BA} \end{bmatrix} = \begin{bmatrix} 0 & 0 \\ 0 & \frac{3EI}{L} \end{bmatrix} \cdot \begin{bmatrix} \phi_{AB} \\ \phi_{BA} \end{bmatrix} \quad (7)$$

Similarly, if a hinge were to exist at the end B as in Fig. 3(c),

$$\begin{bmatrix} M_{AB} \\ M_{BA} \end{bmatrix} = \begin{bmatrix} \frac{3EI}{L} & 0 \\ 0 & 0 \end{bmatrix} \cdot \begin{bmatrix} \phi_{AB} \\ \phi_{BA} \end{bmatrix} \quad (8)$$

By adopting a numerical system rather than an alphabetic system for identifying the ends of each member, with the odd number always smaller than the even number, Wang was able to achieve the necessary changes to the matrix (S) in accordance with Eqs. (7) and (8) using the computed location of any hinge. For example, the substitution of a hinge at a

position 16 in any frame would necessitate the following alterations to the (S) matrix.

$$s^1 (15,15) = \frac{3}{4} S(15,15)$$

$$s^1 (15,16) = s^1 (16,16) = s^1(16,15) = 0$$

where the primes denote the new values. . If a hinge occurred at position 15,

$$s^1 (15,15) = s^1 (15,16) = s^1(16,15) = 0$$

$$s^1 (16,16) = \frac{3}{4} S(16,16)$$

Simple tests exist in computer languages for detecting whether a number is odd or even and then the appropriate changes to the member stiffness matrix (S) can be made.

#### IV DESCRIPTION OF PROGRAM

The Fortran program is included in the Appendix A and the principal stages in its operation are shown in the flow diagram in Fig. 4. The first step is to dimension the arrays and it should be understood that the program will analyze frames whose arrays cannot exceed the initially set sizes but which can be of any size smaller than the initially set values. A discussion of the limitations on frames sizes that can be accommodated by a given machine will be given in the next section. The first item of data must be the identifying number of the frame which if negative, is regarded as the exit signal. Next, the degree of freedom  $\bar{L}$  and the member count NM are read and these two numbers will control the sizes of all the subsequent arrays built within the store for the frame being studied. All the elements of the statics matrix are then read in row by row. This is followed by the member stiffness and plastic moment data with one card per member containing the  $EI/L$  and  $M_p$  values. From this information, the member stiffness matrix (S) and the plastic moment vector will be constructed in the store. The (S) matrix is output for checking, together with headings and the full statics matrix (A). Wang's original program has been modified considerably in this region by incorporating the ability to analyze the same frame for a series of different loading conditions. Accordingly, the next item input has to be the identifying number of the load set which is to follow. For load sets other than the first, the completely deteriorated member stiffness matrix is reconstructed before the analysis proceeds. If the load set number is negative, the program will look for data for a new frame and if no further frames are to be studied, the final card has to contain a negative integer

in the place of a frame identifying number. Hence, the final two cards in any run will contain negative integers. With the load set input and printed for checking purposes, the program proceeds with the analysis by building the frame stiffness matrix (K) from the member stiffness matrix (S) and the statics matrix (A) according to equation (5). The equations are solved for deformations and if these are too large, an indication is given that the frame has reached the collapse condition. Deformations exceeding the value of  $10^4$  are regarded as being too large. If this is not the case, the moments are computed using Eqs. (2) and (4) and the smallest load factor sought so that the position of a plastic hinge can be found.

This part of the program follows Wang's original scheme except for one alteration. It was found that erroneous results were produced for some frames by the original program because the load factors were computed by dividing the residual moment capacity by the absolute value of the moments caused by unit loading. Such a procedure is satisfactory provided the unit load moments at the critical positions are of the same sign in the successive analysis of the frames of deteriorated stiffness. It may well be the case that the moment at the position with the least reserve of strength may be decreasing under increasing load. A test has been incorporated in the section of the program concerned with the finding of the smallest load factor to determine whether such is the case and if so, the position in question is not included in the search for the smallest load factor.

In his program, Wang incorporated four separate tests to determine whether the collapse load for a frame had been reached. One of those tests involved the minimum load factor which, if too small, would indicate that

the load-deflection curve for a frame was close to horizontal. However, it was found that this test would frequently terminate prematurely the analysis for any frame where two plastic hinges might form simultaneously. This test has been omitted from the present program since it is considered that a deformation limitation will determine effectively whether or not the load-displacement curve might be horizontal. Only one of the other two tests for collapse which were provided by Wang has been retained. This is the test which outputs the message "division by zero in inversion" and it effectively determines the stage at which a row and column in the frame stiffness matrix (K) contains only zero terms. In theory, this is the only necessary test but the other is required because rounding-off errors in the floating point arithmetic could delay the program termination and invalidate the final calculation of plastic hinge rotations.

After the mechanism of failure has been found and the load factor and the cumulative moments and deformations at the maximum load are printed, the final computation concerns the amount of plastic rotation that would have occurred at the positions of all plastic hinges except the last formed. Referring to equation (2), the relative end slopes ( $\phi$ ) could be calculated by pre-multiplying the list of cumulative moments (SR) by the inverted form of the member stiffness matrix.

$$(\phi) = (S)^{-1} \cdot (SR) \quad (9)$$

It can be noted that a simple inversion of (S) is not possible because at the final stages of an analysis, this matrix does not exist in its original form. Wang inverted (S) and stored the data at the beginning of his program but was aware of the fact that considerable economy of storage capacity would result if the elements of (S) were stored as a list and the matrix reconstructed in its actual or inverted form when required. This has been



done in the present program.

The slopes computed from equation (9) will be the same as those which can be calculated from equation (4) only at those positions where no plastic deformation has occurred. Accordingly, the amount of plastic hinge rotation can be expressed,

$$(\phi_P) = (S^{-1}) \cdot (SR) - (A^T) \cdot (X) \quad (10)$$

where the lists (SR) and (X) are the moments and deformations in the frame at the stage when the last plastic hinge has just been formed.

Finally, control is returned to see if any further load sets are to be studied for the frame in question. If so, the member stiffness matrix would need to be completely reconstructed as it would have been altered considerably in the course of the analysis for the first load set. If no further load sets are available, the program will commence the analysis of another problem. If there are no further frames to be studied, the run will terminate.

## V PROGRAM LIMITATIONS

The program will perform a first-order elastic-plastic analysis of rigid planes of prismatic members and in its present form is strictly limited to this form of analysis. Since axial stiffness of members is assumed to be infinite, the axial forces present in the members are not calculated explicitly so that it is not possible to arrange for a progressive decrease in plastic moment capacity caused by the presence of axial load. However, it is always possible to account approximately for this effect by reading initial values for plastic moments, already reduced by the estimated axial loads at failure. To account explicitly for axial strains, the member stiffness matrix would consist of (3 x 3) units for each member instead of the (2 x 2) units currently specified so that for a limited computer store capacity, the size of frame to be handled would be curtailed drastically. To account for second-order effects in the displacement analysis, the axial forces in members would be needed with the capacity disadvantage mentioned above, but then the reduction in stiffness of each member could be readily computed and the member stiffness matrix modified progressively in essentially iterative solution procedure. Running time would increase greatly as a result.

The statics matrix also would require progressive modification to account for sway deformations and whereas programs can always be written to do this for any specific frame, it is difficult to visualize a general program that could account for the phenomenon for any type of rigid frame. The great advantage of Wang's scheme is that it can be used for any type of plane frame as a standard program.

The main limiting factor in the use of a general program for frame analysis is storage capacity since the use of matrix methods has the disadvantage that quite extensive arrays can be generated by only moderately sized structures. It is evident that methods can always be developed to utilize tapes as a backing store for a specific machine but the generality of a program is then lost. It is anticipated that core store capacities of computers of the next generation will be greatly in excess of those currently available, so that it will be possible to analyze with an elastic-plastic program the range of sizes of steel frames for which such an analysis is currently relevant.

In its original form, Wang's program required a storage capacity which can be expressed,

$$C \geq (\bar{L} + 2NM)^2 + 4NM^2 + 3\bar{L} + 14NM \quad (11)$$

where  $C$  = capacity,

$\bar{L}$  = degree of freedom, and

$NM$  = the number of members in a frame

As has been explained, there is no need to store the inverted form of the member stiffness matrix if this can be generated when required from a one dimensional list of member stiffness parameters. The capacity required for the modified program can be expressed,

$$C \geq (\bar{L} + 2NM)^2 + 3\bar{L} + 15NM \quad (12)$$

As an example, a three-story, two bay rigid frame subjected to two-point loading on each beam would have 36 degrees of freedom and 27 members so that the original program would require a capacity of 11502 locations. The modified program would require the reduced capacity of 8613 locations. (The data capacity of the Lehigh GE 225 computer when using the elastic-plastic program was found to be 1860 whereas with an IBM 7074

machine, the capacity was 6850 locations.) It is apparent that load application positions have to be treated as joints so that a beam under two-point loading constitutes three members. Consequently, the available capacity of a medium sized machine such as the GE 225 will be fully utilized by frames of only moderate size.

One further limiting factor should be mentioned. It can sometimes occur in steel frames that a plastic hinge which is formed early in the loading history may not be required in the collapse condition. The moment at such a section would decrease in magnitude and a plastic hinge would not then exist. This phenomenon cannot be accounted for in the present program as the process of free hinge insertion is irreversible. The calculated load factor for such a problem would err on the safe side since the equilibrium and yield conditions would be satisfied but not the mechanism condition. This phenomenon has been mentioned by Finzi<sup>(7)</sup>. The example of a two-span beam, which has been used by Neal<sup>(8)</sup> to demonstrate this phenomenon, is shown in Fig. 5. For the loads shown at (a), an elastic analysis will produce a maximum moment at the point(4) as can be seen in (b). However, a simple plastic analysis will predict a failure mechanism with plastic hinges at (3) and (6) but not at (4). This can be deduced from the moment diagram shown at (d) in the figure. The results obtained from a computer analysis of this problem are in the Appendix C. It can be seen that the computer correctly detects the formation of the first hinge at position (4) and the second at (6) as shown at (c) in Fig. 5 but cannot account for the closing of the first formed hinge thereafter. Accordingly, it arrives at an invalid collapse mechanism with a load factor smaller than the correct one. Consequently, it is desirable for any frame to check the collapse mechanism arrived at by the computer to see whether or

not it is valid.

A related problem is that of the formation of a plastic hinge under a distributed load. In such a case, the loading must be replaced by equivalent point loads, as many being chosen as the computer capacity will accommodate.

## VI CONCLUSION

The Wang program is a very powerful tool in the analysis and design of steel structures and has been used to study the economics of steel frame design using the various grades of high tensile steel currently available. Any such study would evidently involve the analysis of many trial designs and, for frames other than simple one story portals, the computational problem would be insuperable without the use of a computer program such as the one described. As a final example, the frame analyzed by Heyman<sup>(6)</sup> is shown in Fig. 6 and a selection of pages from the computer output is shown in the Appendix D. The computed load-sway curve is shown in Fig. 7. The complete print out for this frame consisted of over 40 pages and the total time for both compilation and execution on an IBM 7074 was less than 3 minutes. The preparation of the statics matrix which is the collection of all the equations of equilibrium for the structure was a simple matter taking less than half an hour. The print out of this matrix is also shown in the Appendix D. A more detailed explanation of statics matrices has been given elsewhere.<sup>(9)</sup>

The sign convention adopted in the solution of Heyman's problem is that in which clockwise moments acting on the ends of members are regarded as positive together with downward vertical loads. The program itself is not dependent upon any particular sign convention and will operate successfully as long as a self-consistent convention is adopted in the statics matrix and in the vector of applied loads.

## VII ACKNOWLEDGEMENTS

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At the time of writing, the author was on study leave from the University of Sydney, Australia.

NOMENCLATURE

(A)	the statics matrix
$(A^T)$	the statics matrix transposed ( = (C) )
(C)	the kinematics matrix
C	computer capacity for data
E	Young's modulus
I	second moment of area
I, J	identifying integers
(K)	the frame stiffness matrix
L	length of a prismatic member
$\bar{L}$	degree of freedom
NM	count of the members in a frame
$M_p$	full plastic moment of resistance
$M_{AB}$	moment applied at end A of member AB
(S)	the member stiffness matrix for a frame
$(S_{AB})$	member stiffness matrix for member AB
(SR)	stress resultant vector for a frame
$(SR_{AB})$	stress resultant vector for member AB
SLG	smallest load factor
$S(I, J)$	an element in the member stiffness matrix
$S'(I, J)$	a new value for $S(I, J)$
$(S)^{-1}$	the inverted member stiffness matrix
(W)	the applied load vector
W	an applied point load
(X)	the frame deformation vector
$(\phi_{AB})$	relative deformation vector for member AB



- ( $\emptyset$ ) relative deformation vector for a frame
- ( $\emptyset_p$ ) plastic hinge rotation vector

IX APPENDIXESAppendix AThe Fortran Program

The statements of the program are contained in the following pages. (pp. 25 - 31) They are shown in the form used by the IBM 7074 computer but the only changes necessary for the GE 225 machine are the following substitutions:

READ       for       READ INPUT TAPE 1,  
and  
PRINT       for       WRITE OUTPUT TAPE 2,

Format requirements in Fortran impose some limitations on the choice of names for variables but as far as possible, the names used correspond with those used in the text. The identification of the principal variables used in the program is shown in Table II.

Table II Identification of Variables

Variable	In Text	In Program
the statics matrix	(A)	A(I,J)
load factors	-	ALG(I)
frame stiffness matrix	(K)	ASAT(I,J)
frame deformation vector	(X)	ASAT(I,L+1)
relative deformation vector	$(A^T) \cdot (X) = (\phi)$	ATX(I)
cumulative load factor	-	CLG
cumulative moment vector	-	CM(I)
relative deformation vector	$(S^{-1}) (SR) = (\phi)$	DM(I)
plastic hinge rotation vector	$(\phi_p)$	H(I)
frame identification number	-	JJ
load set identification number	-	KK
location of plastic hinge	-	NPH
analysis stage number	-	NCYCL
the applied unit load vector	(W)	P(I)
initial plastic moment vector	-	PM(I)
the member stiffness matrix	(S)	S(I,J)
an intermediate matrix product	$(S) \cdot (A^T)$	SAT(I,J)
smallest load factor	-	SALG
moments caused by unit loads	(SR)	SATX(I)
member stiffness data vector	-	SDAT(I)

C		0001
C		0002
C	LEHIGH UNIVERSITY FRITZ LABORATORY APRIL, 1965.	0003
C		0004
C	FIRST ORDER ELASTIC PLASTIC PLANE FRAME ANALYSIS.	0005
C		0006
C	BASIC PROGRAM BY C.K.WANG, UNIV. OF WISCONSIN, 1963.	0007
C		0008
C	MODIFIED FOR IBM 7074 BY H.B.HARRISON.	0009
C		0010
C		0011
C		0012
C	<u>SPECIFY THE MAXIMUM SIZES OF ALL THE MATRICES.</u>	0013
C		0014
	DIMENSION A(30,48), S(48,48), SAT(48,30), SATX(48)	0015
	DIMENSION P(30), ASAT(30,31), PM(48), ALG(48)	0016
	DIMENSION CX(30), CM(48), SDAT(24), UM(48)	0017
	DIMENSION ATX(48), H(48)	0018
C		0019
C		0020
C	<u>INPUT FRAME NUMBER, EXIT IF NEGATIVE.</u>	0021
C		0022
	9 READ INPUT TAPE 1, 13, JJ	0023
	13 FORMAT (I15)	0024
	IF (JJ) 99,5,5	0025
C		0026
C		0027
C	<u>INPUT THE DEGREES OF FREEDOM L, MEMBER COUNT NM.</u>	0028
C		0029
	5 READ INPUT TAPE 1, 23, L, NM	0030
	23 FORMAT (2I5)	0031
	M = 2 * NM	0032
C		0033
C		0034
C	<u>INPUT ALL ELEMENTS OF THE STATICS MATRIX A.</u>	0035
C		0036
	READ INPUT TAPE 1, 35, ((A(I,J), J=1,M), I=1,L)	0037
	35 FORMAT(7F10,4)	0038
C		0039
C		0040
C	<u>INPUT THE MEMBER PROPERTIES, EI/L AND Mp.</u>	0041
C		0042
	DO 166 I = 1,NM	0043
	READ INPUT TAPE 1, 167, SDAT(I), PMI	0044
	K = 2*I	0045
	PM(K) = PMI	0046
	PM(K-1) = PM(K)	0047
	166 CONTINUE	0048
	167 FORMAT(2F10,4)	0049
C		0050
C		0051
C	<u>BUILD THE MEMBER STIFFNESS MATRIX S.</u>	0052
C		0053
	DO 160 I = 1,M	0054
	DO 160 J = 1,M	0055

## Appendix A Fortran Statements

160	S(I,J) = 0.	0056
	DO 161 I = 1,M	0057
	ITEST = I/2*2-1	0058
	IF(ITEST) 162,161,161	0059
162	K = I/2 + 1	0060
	S(I,I) = 4.0 * SDAT(K)	0061
	S(I+1,I+1) = S(I,I)	0062
	S(I,I+1) = 0.5 * S(I,I)	0063
	S(I+1,I) = S(I,I+1)	0064
161	CONTINUE	0065
C		0066
C		0067
C	<u>OUTPUT TITLES, THE STATICS MATRIX A, STIFFNESS MATRIX S.</u>	0068
C		0069
	WRITE OUTPUT TAPE 2, 97, JJ	0070
97	FORMAT(150H1ELASTIC PLASTIC FIRST ORDER ANALYSIS OF FRAME NO.,I3/)	0071
	WRITE OUTPUT TAPE 2, 7	0072
7	FORMAT(21H0THE STATICS MATRIX//)	0073
	DO 1 I=1,L	0074
1	WRITE OUTPUT TAPE 2, 21,I, (A(I,J), J = 1,M)	0075
21	FORMAT(4H0R0H,I3,1X, 7E16.7/(8X,7E16.7))	0076
	WRITE OUTPUT TAPE 2, 17	0077
17	FORMAT(23H1THE STIFFNESS MATRIX//)	0078
	DO 2 I=1,M	0079
2	WRITE OUTPUT TAPE 2, 21, I, (S(I,J), J = 1,M)	0080
C		0081
C	INPUT THE LOAD SET NO. KK,IF NEG. COMMENCE THE NEXT FRAME.	0082
C		0083
708	READ INPUT TAPE 1, 13, KK	0084
	IF(KK-1) 9,707,800	0085
C		0086
C	IF KK IS UNITY, BYPASS THE NEXT BLOCK OF INSTRUCTIONS.	0087
C	<u>IF KK IS GREATER THAN UNITY, REBUILD THE S MATRIX.</u>	0088
C		0089
800	DO 901 I = 1,M	0090
	DO 901 J = 1,M	0091
	S(I,J) = 0.	0092
901	CONTINUE	0093
	DO 931 I = 1,M	0094
	ITEST = (I/2*2) - 1	0095
	IF (ITEST) 932,931,931	0096
932	K = I/2 + 1	0097
	S(I,I) = 4.0 * SDAT(K)	0098
	S(I+1,I+1) = S(I,I)	0099
	S(I,I+1) = 0.5 * S(I,I)	0100
	S(I+1,I) = S(I,I+1)	0101
931	CONTINUE	0102
C		0103
707	WRITE OUTPUT TAPE 2, 27, KK	0104
27	FORMAT(33H1THE EXTERNAL LOAD VECTOR SET NO., I3)	0105
C		0106
C		0107
C	<u>INPUT THE LOAD SET VECTOR P.</u>	0108
C		0109
	READ INPUT TAPE 1, 35,(P(I), I = 1,L)	0110

	DO 3 I=1,L	0111
	3 WRITE OUTPUT TAPE 2, 21, 1, P(I)	0112
C		0113
C		0114
C	<u>SET TO ZERO THE VARIABLES NCYCL, CLG AND THE ARRAYS CX, CM, SAT,</u>	0115
		0116
	NCYCL = 0	0117
	CLG = 0.	0118
	DO 24 I=1,L	0119
24	CX(I) = 0.	0120
	DO 24 I=1,M	0121
24	CM(I) = 0.	0122
		0123
		0124
C		0125
C	<u>BE ENTRY POINT FOR SUCCESSIVE ANALYSES OF DETERIORATED FRAMES,</u>	0126
		0127
15	DO 15 I=1,M	0128
	DO 15 J=1,L	0129
	SAT(I,J)=0.	0130
		0131
C		0132
C	<u>POST MULTIPLY S BY TRANSPOSED A TO GET MATRIX SAT,</u>	0133
		0134
	DO 20 K=1,M	0135
20	SAT(I,J)=SAT(I,J)+S(I,K)*A(J,K)	0136
10	CONTINUE	0137
		0138
C		0139
C	<u>PREMULTIPLY SAT BY A TO GET MATRIX ASAT,</u>	0140
		0141
	DO 30 I=1,L	0142
	DO 30 J=1,L	0143
	ASAT(I,J)=0.	0144
	DO 40 K=1,M	0145
40	ASAT(I,J)=ASAT(I,J)+A(I,K)*SAT(K,J)	0146
30	CONTINUE	0147
		0148
C		0149
C	<u>SOLVE THE SLOPE DEFLECTION EQUATIONS FOR THE GIVEN LOAD SET,</u>	0150
	<u>STORE THE SOLUTION IN THE LAST COLUMN OF THE MATRIX ASAT.</u>	0151
		0152
	LP1=L+1	0153
	DO 50 I=1,L	0154
50	ASAT(I,LP1)=P(I)	0155
	DO 60 I=1,L	0156
	IP1=I+1	0157
	TEMP=ABSF(ASAT(I,I))	0158
	K=I	0159
	DO 61 J=I,L	0160
	IF (ABSF(ASAT(J,I))-TEMP) 61,61,62	0161
62	K=J	0162
	TEMP = ABSF(ASAT(J,I))	0163
61	CONTINUE	0164
	IF (K-I) 72,71,72	0165
72	DO 45 J=I,IP1	

	TEMP = ASAT(I,J)	0166
	ASAT(I,J) = ASAT(K,J)	0167
	45 ASAT(K,J) = TEMP	0168
	71 IF (ASAT(I,I)) 16,147,16	0169
	147 WRITE OUTPUT TAPE 2, 347	0170
	347 FORMAT (30H0DIVISION BY ZERO IN INVERSION)	0171
	GO TO 47	0172
	16 TEMP = 1./ASAT(I,I)	0173
	DO 70 J=1,LP1	0174
	70 ASAT(I,J)=ASAT(I,J)*TEMP	0175
	DO 60 J=1,L	0176
	IF (I-J) 59,60,59	0177
	59 TEMP=ASAT(J,I)	0178
	DO 80 K=IP1,IP1	0179
	80 ASAT(I,K)=ASAT(I,K)-TEMP*ASAT(I,K)	0180
	60 CONTINUE	0181
C		0182
C		0183
C	<u>PRINT THE DEFORMATIONS DUE TO UNIT LOADS, TEST IF TOO LARGE.</u>	0184
C		0185
	WRITE OUTPUT TAPE 2, 511	0186
	511 FORMAT (30H0REFLECTIONS DUE TO UNIT LOADS)	0187
	DO 51 I=1,L	0188
	51 WRITE OUTPUT TAPE 2, 21, I, ASAT(I,LP1)	0189
	DO 311 I=1,L	0190
	TEMP = ARSF[ASAT(I,LP1)] - 1.E+04	0191
	IF (TEMP) 311,647,647	0192
	311 CONTINUE	0193
	GO TO 303	0194
	647 WRITE OUTPUT TAPE 2, 847	0195
	647 FORMAT (21H0DEFLECTION TOO LARGE)	0196
	GO TO 47	0197
C		0198
C		0199
C	<u>COMPUTE AND OUTPUT MOMENTS DUE TO THE UNIT LOADS:</u>	0200
C		0201
	303 DO 110 I=1,M	0202
	SATX(I) = 0.	0203
	DO 120 K=1,L	0204
	120 SATX(I)=SATX(I)+SAT(I,K)*ASAT(K,LP1)	0205
	110 CONTINUE	0206
	WRITE OUTPUT TAPE 2, 522	0207
	522 FORMAT (26H0MOMENTS DUE TO UNIT LOADS)	0208
	DO 52 I=1,M	0209
	52 WRITE OUTPUT TAPE 2, 21, I, SATX(I)	0210
C		0211
C		0212
C	<u>CALCULATE THE LOAD FACTOR ALG AT EACH END OF EACH MEMBER.</u>	0213
C		0214
	DO 201 I=1,M	0215
	IF(ARSF[SATX(I)] - 1.0) 202,202,203	0216
	202 ALG(I) = 1.E20	0217
	GO TO 201	0218
	203 ALG(I) = (PM(I)-ARSF[CM(I)]) / ARSF[SATX(I)]	0219
	201 CONTINUE	0220

C		0221
C		0222
C	<u>FIND POSITION AND VALUE OF SMALLEST LOAD FACTOR SALG.</u>	0223
C		0224
	SALG = 1.E20	0225
	DO 204 I = 1,M	0226
	TEST = CM(I) * SATX(I)	0227
	IF (TEST) 204,205,205	0228
	205 IF(ALG(I) - SALG) 1206,204,204	0229
	1206 SALG = ALG(I)	0230
	NPH = I	0231
	204 CONTINUE	0232
C		0233
C		0234
C	<u>FACTOR UNIT MOMENTS BY SALG AND GET CUMULATIVE MOMENTS.</u>	0235
C		0236
	302 DO 207 I=1,M	0237
	SATX(I) = SALG*SATX(I)	0238
	207 CM(I) = CM(I) + SATX(I)	0239
C		0240
C		0241
C	<u>CALCULATE THE CUMULATIVE LOAD FACTOR. CLG.</u>	0242
C		0243
	304 CLG = CLG + SALG	0244
C		0245
C		0246
C	<u>MULTIPLY UNIT DEFLECTIONS BY SMALLEST LOAD FACTOR SALG.</u>	0247
C		0248
	DO 206 I=1,L	0249
	ASAT(I,LP1) = SALG*ASAT(I,LP1)	0250
C		0251
C		0252
C	<u>CALCULATE CUMULATIVE DEFLECTIONS.</u>	0253
C		0254
	206 CX(I) = CX(I) + ASAT(I,LP1)	0255
C		0256
C	<u>UPDATE THE CYCLE NUMBER NCYCL.</u>	0257
C		0258
	NCYCL = NCYCL + 1	0259
C		0260
C		0261
C	<u>OUTPUT CYCLE NO. AND LOCATION OF PLASTIC HINGE, LOAD FACTOR ETC</u>	0262
C		0263
	WRITE OUTPUT TAPE 2, 401, NCYCL, NPH	0264
	401 FORMAT(18H1PLASTIC HINGE NO., I3, 2X, 15HFORMED AT POINT, I3, I)	0265
	WRITE OUTPUT TAPE 2, 402	0266
	402 FORMAT(12H0LOAD FACTOR, 3X, 10HADDITIONAL, 9X, 10HCUMULATIVE)	0267
	WRITE OUTPUT TAPE 2, 403, NCYCL, SALG, CLG	0268
	403 FORMAT(16H0STAGE, I4, E18.7, E18.7)	0269
C		0270
C		0271
C	<u>OUTPUT ADDITIONAL AND CUMULATIVE DEFLECTIONS.</u>	0272
C		0273
	WRITE OUTPUT TAPE 2, 404	0274
	404 FORMAT(12H0DEFORMATION, 3X, 10HADDITIONAL, 9X, 10HCUMULATIVE/)	0275



```

DO 208 I=1,L                                0276
208 WRITE OUTPUT TAPE 2, 405, I, ASAT(I,LP1), CX(I) 0277
405 FORMAT (3H A1, I4, E22.7, F19.7)          0278
C                                             0279
C                                             0280
C           OUTPUT ADDITIONAL AND CUMULATIVE MOMENTS WITH PL.MOMENT CAP. 0281
C                                             0282
WRITE OUTPUT TAPE 2, 405                     0283
406 FORMAT(7H0MOMENTRY,10HADDITIONAL,9X10HCUMULATIVE10X,8HPLAS MOM/) 0284
DO 209 I=1,M                                  0285
209 WRITE OUTPUT TAPE 2, 407, I, SATX(I), CM(I), PM(I) 0286
407 FORMAT (3H A1, I4, F18.4,2F19.4)         0287
C                                             0288
C                                             0289
C           CHANGE THE STIFFNESS MATRIX ACCORDING TO WHERE THE LAST 0290
C           PLASTIC HINGE WAS FOUND.          0291
C                                             0292
ITEST = (NPH/2 * 2) - NPH                    0293
IF (ITEST) 211,210,210                       0294
210 S(NPH-1,NPH-1)=0.75*S(NPH-1,NPH-1)      0295
S(NPH,NPH) = 0.                               0296
S(NPH-1,NPH) = 0.                               0297
S(NPH,NPH-1) = 0.                               0298
GO TO 212                                       0299
211 S(NPH+1,NPH+1) = 0.75*S(NPH+1,NPH+1)    0300
S(NPH,NPH) = 0.                               0301
S(NPH,NPH+1) = 0.                               0302
S(NPH+1,NPH) = 0.                               0303
C                                             0304
C                                             0305
C           RETURN CONTROL TO ANALYSE THE DETERIORATED FRAME. 0306
C                                             0307
212 GO TO 15                                    0308
C                                             0309
C                                             0310
C           COMPUTE THE HINGE ROTATIONS ONCE THE COLLAPSE MECHANISM HAS 0311
C           BEEN FOUND. FIRST, INVERT THE S MATRIX.          0312
C                                             0313
47 WRITE OUTPUT TAPE 2, 408                   0314
408 FORMAT (36H0COLLAPSE MECHANISM HAS BEEN REACHED) 0315
DO 163 I = 1,M                                0316
DO 163 J = 1,M                                0317
163 S(I,J) = 0.                                0318
DO 164 I = 1,M                                0319
ITEST = I/2*2-1                               0320
IF(ITEST) 165,164,164                         0321
165 K = I/2 + 1                               0322
S(I,I) = 1.0/(3.0 * SDAT(K))                  0323
S(I+1,J+1) = S(I,I)                          0324
S(I,I+1) = -0.5*S(I,I)                       0325
S(I+1,I) = S(I,I+1)                          0326
164 CONTINUE                                  0327
DO 163 I=1,M                                  0328
DM(I) = 0.                                     0329
DO 164 K=1,M                                  0330

```

134	DM(I) = DM(I) + S(I,K) * CM(K)	0331
134	CONTINUE	0332
	DO 135 I=1,M	0333
	ATX(I) = 0.	0334
	DO 136 K=1,L	0335
136	ATX(I) = ATX(I) + A(K,I)*CX(K)	0336
135	CONTINUE	0337
	DO 137 I=1,M	0338
137	H(I) = UM(I) - ATX(I)	0339
	WRITE OUTPUT TAPE 2, 138	0340
138	FORMAT (1H0,14X,15HHINGE ROTATIONS/)	0341
	DO 139 I=1,M	0342
139	WRITE OUTPUT TAPE 2, 140, I, H(I)	0343
140	FORMAT (9H AT PCINT, 14,E15.7)	0344
C		0345
C		0346
C	<u>RETURN CONTROL TO SEE IF ANY MORE LOAD SETS.</u>	0347
C		0348
	GO TO 708	0349
C		0350
99	CALL EXIT	0351
	END	0352

ELASTIC PLASTIC FIRST ORDER ANALYSIS OF FRAME NO. 2

PROPPED CANTILEVER PROBLEM

THE STATICS MATRIX

ROW 1	0.1000000E 01	0.0000000E 00	0.0000000E 00	0.0000000E 00
ROW 2	0.0000000E 00	0.1000000E 01	0.1000000E 01	0.0000000E 00
ROW 3	-0.2000000E 00	-0.2000000E 00	0.2000000E 00	0.2000000E 00

THE STIFFNESS MATRIX

ROW 1	0.4000000E 04	0.2000000E 04	0.0000000E 00	0.0000000E 00
ROW 2	0.2000000E 04	0.4000000E 04	0.0000000E 00	0.0000000E 00
ROW 3	0.0000000E 00	0.0000000E 00	0.4000000E 04	0.2000000E 04
ROW 4	0.0000000E 00	0.0000000E 00	0.2000000E 04	0.4000000E 04

THE EXTERNAL LOAD VECTOR SET NO. 1

ROW 1	0.0000000E 00
ROW 2	0.0000000E 00
ROW 3	0.1000000E 01

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.6250000E-03
ROW 2	-0.1562500E-03
ROW 3	0.1822917E-02

MOMENTS DUE TO UNIT LOADS

ROW 1	-0.1000000E-06
ROW 2	-0.1562500E 01
ROW 3	0.1562500E 01
ROW 4	0.1875000E 01

Appendix B Solution to Propped Cantilever

PLASTIC HINGE NO. 1 FORMED AT POINT 4

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 1	0.2133333E 02	0.2133333E 02	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.1333333E-01	0.1333333E-01	
AT 2	-0.3333333E-02	-0.3333333E-02	
AT 3	0.3888889E-01	0.3888889E-01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
AT 1	-0.0000	-0.0000	40.0000
AT 2	-33.3333	-33.3333	40.0000
AT 3	33.3333	33.3333	40.0000
AT 4	40.0000	40.0000	40.0000

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.1250000E-02
ROW 2	0.0000000E 00
ROW 3	0.4166667E-02

MOMENTS DUE TO UNIT LOADS

ROW 1	-0.3000000E-06
ROW 2	-0.2500000E 01
ROW 3	0.2500000E 01
ROW 4	0.0000000E 00

PLASTIC HINGE NO. 2 FORMED AT POINT 2

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 2	0.2666667E 01	0.2400000E 02	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.3333333E-02	0.1666667E-01	
AT 2	0.0000000E 00	-0.3333333E-02	
AT 3	0.1111111E-01	0.5000000E-01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
AT 1	-0.0000	-0.0000	40.0000
AT 2	-6.6667	-40.0000	40.0000
AT 3	6.6667	40.0000	40.0000
AT 4	0.0000	40.0000	40.0000

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.1000000E 05
ROW 2	-0.1000000E 05
ROW 3	0.5000000E 05

DEFLECTION TOO LARGECOLLAPSE MECHANISM HAS BEEN REACHEDHINGE ROTATIONS

AT POINT 1	-0.8000000E-09
AT POINT 2	0.2000000E-08
AT POINT 3	-0.1000000E-08
AT POINT 4	-0.3333334E-02

Appendix B (cont.)

ELASTIC PLASTIC FIRST ORDER ANALYSIS OF FRAME NO. 13

NEAL - FINZI PROBLEM

THE STATICS MATRIX

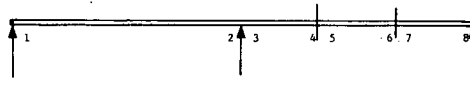
ROW 1	0.000000E 00 0.000000E 00	0.000000E 00	-0.100000E 01	-0.100000E 01	0.100000E 01	0.100000E 01	0.000000E 00
ROW 2	0.000000E 00 0.100000E 01	0.000000E 00	0.000000E 00	0.000000E 00	-0.100000E 01	-0.100000E 01	0.100000E 01
ROW 3	0.100000E 01 0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00
ROW 4	0.000000E 00 0.000000E 00	0.100000E 01	0.100000E 01	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00
ROW 5	0.000000E 00 0.000000E 00	0.000000E 00	0.000000E 00	0.100000E 01	0.100000E 01	0.000000E 00	0.000000E 00
ROW 6	0.000000E 00 0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.100000E 01	0.100000E 01
ROW 7	0.000000E 00 0.100000E 01	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00

THE STIFFNESS MATRIX

ROW 1	0.133333E 01 0.000000E 00	0.666667E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00
ROW 2	0.666667E 00 0.000000E 00	0.133333E 01	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00
ROW 3	0.000000E 00 0.000000E 00	0.000000E 00	0.400000E 01	0.200000E 01	0.000000E 00	0.000000E 00	0.000000E 00
ROW 4	0.000000E 00 0.000000E 00	0.000000E 00	0.200000E 01	0.400000E 01	0.000000E 00	0.000000E 00	0.000000E 00
ROW 5	0.000000E 00 0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.400000E 01	0.200000E 01	0.000000E 00
ROW 6	0.000000E 00 0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.200000E 01	0.400000E 01	0.000000E 00
ROW 7	0.000000E 00 0.200000E 01	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.400000E 01
ROW 8	0.000000E 00 0.400000E 01	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.000000E 00	0.200000E 01

THE EXTERNAL LOAD VECTOR SET NO. 1

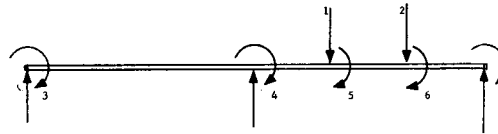
ROW 1	0.130000E 01
ROW 2	0.700000E 00
ROW 3	0.000000E 00
ROW 4	0.000000E 00
ROW 5	0.000000E 00
ROW 6	0.000000E 00
ROW 7	0.000000E 00



MEMBER IDENTIFICATION SEQUENCE

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.562962E 00
ROW 2	0.587036E 00
ROW 3	-0.258333E 00
ROW 4	0.516666E 00
ROW 5	0.397222E 00
ROW 6	-0.344443E 00
ROW 7	-0.708333E 00



DEFORMATION IDENTIFICATION SEQUENCE

MOMENTS DUE TO UNIT LOADS

ROW 1	0.200000E-07
ROW 2	0.516666E 00
ROW 3	-0.516666E 00
ROW 4	-0.755552E 00
ROW 5	0.755554E 00
ROW 6	-0.727777E 00
ROW 7	0.727777E 00
ROW 8	-0.100000E-06

NEAL - FINZI PROBLEM

Appendix C Solution to Neal-Finzi Problem

PLASTIC HINGE NO. 1 FORMED AT POINT 4

PLASTIC HINGE NO. 2 FORMED AT POINT 6

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 1	0.1323530E 01	0.1323530E 01	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.7450979E 00	0.7450979E 00	
AT 2	0.7769607E 00	0.7769607E 00	
AT 3	-0.3419117E 00	-0.3419117E 00	
AT 4	0.6838235E 00	0.6838235E 00	
AT 5	0.5257352E 00	0.5257352E 00	
AT 6	-0.4558823E 00	-0.4558823E 00	
AT 7	-0.9374999E 00	-0.9374999E 00	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
AT 1	0.0000	0.0000	1.0000
AT 2	0.6838	0.6838	1.0000
AT 3	-0.6838	-0.6838	1.0000
AT 4	-1.0000	-1.0000	1.0000
AT 5	1.0000	1.0000	1.0000
AT 6	-0.9632	-0.9632	1.0000
AT 7	0.9632	0.9632	1.0000
AT 8	-0.0000	-0.0000	1.0000

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 2	0.1050416E 00	0.1428572E 01	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.2310912E 00	0.9761892E 00	
AT 2	0.1278005E 00	0.9047611E 00	
AT 3	-0.8665922E-01	-0.4285709E 00	
AT 4	0.1733184E 00	0.8571419E 00	
AT 5	-0.9716337E-01	0.4285718E 00	
AT 6	-0.1155456E 00	-0.5714279E 00	
AT 7	-0.1339279E 00	-0.1071428E 01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
AT 1	0.0000	0.0000	1.0000
AT 2	0.1733	0.8571	1.0000
AT 3	-0.1733	-0.8571	1.0000
AT 4	0.0000	-1.0000	1.0000
AT 5	-0.0000	1.0000	1.0000
AT 6	-0.0368	-1.0000	1.0000
AT 7	0.0368	1.0000	1.0000
AT 8	-0.0000	-0.0000	1.0000

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.2199998E 01
ROW 2	0.1216666E 01
ROW 3	-0.8249992E 00
ROW 4	0.1649998E 01
ROW 5	-0.9249992E 00
ROW 6	-0.1099999E 01
ROW 7	-0.1274999E 01

DEFLECTIONS DUE TO UNIT LOADS

ROW 1	0.1820832E 01
ROW 2	0.4375000E 07
ROW 3	-0.7374996E 00
ROW 4	0.1474999E 01
ROW 5	0.4374999E 07
ROW 6	-0.4375000E 07
ROW 7	-0.4375001E 07

MOMENTS DUE TO UNIT LOADS

ROW 1	0.1000000E-06
ROW 2	0.1649998E 01
ROW 3	-0.1649998E 01
ROW 4	0.0000000E 00
ROW 5	-0.1400000E-05
ROW 6	-0.3500009E 00
ROW 7	0.3499996E 00
ROW 8	-0.4000000E-06

DEFLECTION TOO LARGE

COLLAPSE MECHANISM HAS BEEN REACHED

HINGE ROTATIONS

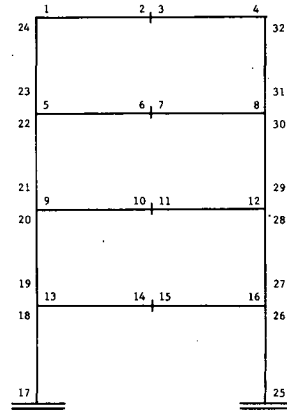
AT POINT 1	0.2000000E-07
AT POINT 2	-0.4000000E-07
AT POINT 3	0.0000000E 00
AT POINT 4	0.3571410E 00
AT POINT 5	-0.4000000E-07
AT POINT 6	-0.1000000E-07
AT POINT 7	-0.2000000E-07
AT POINT 8	-0.8000000E-07



PLASTIC HINGE NO. 1 FORMED AT POINT 16

297.16

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 1	0.1742112E 01	0.1742112E 01	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.7561517E-01	0.7561517E-01	
AT 2	0.532846E-01	0.532846E-01	
AT 3	0.5307721E-01	0.5307721E-01	
AT 4	0.5642525E-01	0.5642525E-01	
AT 5	0.2291726E 00	0.2291726E 00	
AT 6	0.2061622E 00	0.2061622E 00	
AT 7	0.1532738E 00	0.1532738E 00	
AT 8	0.7213038E-01	0.7213038E-01	
AT 9	0.5519973E-02	0.5519973E-02	
AT 10	-0.3615000E-03	-0.3615000E-03	
AT 11	-0.4073974E-02	-0.4073974E-02	
AT 12	0.3911923E-02	0.3911923E-02	
AT 13	-0.1030965E-02	-0.1030965E-02	
AT 14	0.2119359E-03	0.2119359E-03	
AT 15	0.5899208E-02	0.5899208E-02	
AT 16	-0.1801213E-02	-0.1801213E-02	
AT 17	0.1305644E-02	0.1305644E-02	
AT 18	0.7413990E-02	0.7413990E-02	
AT 19	-0.2335401E-02	-0.2335401E-02	
AT 20	0.1927612E-02	0.1927612E-02	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MDM
AT 1	-109.8803	-109.8803	318.3330
AT 2	-255.2833	-255.2833	318.3330
AT 3	255.2833	255.2833	318.3330
AT 4	163.5027	163.5027	318.3330
AT 5	-96.6562	-96.6562	318.3330
AT 6	-218.8554	-218.8554	318.3330
AT 7	218.8554	218.8554	318.3330
AT 8	249.5827	249.5827	318.3330
AT 9	16.8987	16.8987	350.0000
AT 10	-227.3768	-227.3768	350.0000
AT 11	227.3768	227.3768	350.0000
AT 12	312.2976	312.2976	350.0000
AT 13	33.0057	33.0057	350.0000
AT 14	-233.4777	-233.4777	350.0000
AT 15	233.4777	233.4777	350.0000
AT 16	350.0000	350.0000	350.0000
AT 17	-126.8332	-126.8332	213.3330
AT 18	-38.2772	-38.2772	213.3330
AT 19	5.2717	5.2717	213.3330
AT 20	-12.8215	-12.8215	213.3330
AT 21	29.7204	29.7204	213.3330
AT 22	5.9834	5.9834	213.3330
AT 23	90.6731	90.6731	213.3330
AT 24	109.8804	109.8804	213.3330
AT 25	-192.3649	-192.3649	213.3330
AT 26	-169.3407	-169.3407	213.3330
AT 27	-180.6593	-180.6593	213.3330
AT 28	-188.0883	-188.0883	213.3330
AT 29	-124.2090	-124.2090	213.3330
AT 30	-137.2727	-137.2727	213.3330
AT 31	-112.3098	-112.3098	213.3330
AT 32	-163.5026	-163.5026	213.3330

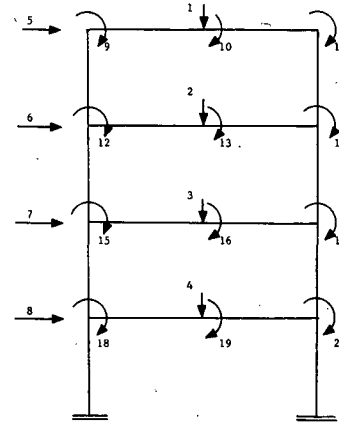


MEMBER IDENTIFICATION SEQUENCE

HEYMAN'S FRAME

PLASTIC HINGE NO. 2- FORMED AT POINT 25

LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 2	0.1608411E 00	0.1902953E 01	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.7022406E-02	0.8263758E-01	
AT 2	0.4815532E-02	0.5832837E-01	
AT 3	0.5488834E-02	0.5856605E-01	
AT 4	0.1484072E-01	0.7126597E-01	
AT 5	0.3892903E-01	0.2681016E 00	
AT 6	0.3667761E-01	0.2428399E 00	
AT 7	0.3078326E-01	0.1840570E 00	
AT 8	0.1452542E-01	0.866559E-01	
AT 9	0.5177058E-03	0.6037679E-02	
AT 10	-0.3466457E-04	-0.3961646E-03	
AT 11	-0.3790476E-03	-0.4453022E-02	
AT 12	0.3630792E-03	0.4275002E-02	
AT 13	-0.1044763E-03	-0.1135441E-02	
AT 14	0.5482599E-04	0.2667619E-03	
AT 15	0.7730907E-03	0.6672299E-02	
AT 16	-0.2412891E-03	-0.2042502E-02	
AT 17	0.1920655E-03	0.1497709E-02	
AT 18	0.1608577E-02	0.9022567E-02	
AT 19	-0.3557223E-04	-0.2370973E-02	
AT 20	0.1875937E-02	0.3803549E-02	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MDM
AT 1	-9.9812	-119.8615	318.3330
AT 2	-23.6370	-278.9204	318.3330
AT 3	23.6370	278.9204	318.3330
AT 4	15.1231	178.6258	318.3330
AT 5	-4.4408	-105.0970	318.3330
AT 6	-19.9998	-238.6552	318.3330
AT 7	19.9998	238.6552	318.3330
AT 8	23.9381	273.5207	318.3330
AT 9	5.6614	-11.2373	350.0000
AT 10	-22.0650	-249.4418	350.0000
AT 11	22.0650	249.4418	350.0000
AT 12	33.9100	346.2076	350.0000
AT 13	5.8339	38.8396	350.0000
AT 14	-39.1062	-272.5839	350.0000
AT 15	39.1062	272.5838	350.0000
AT 16	0.0000	350.0000	350.0000
AT 17	-24.1615	-150.9947	213.3330
AT 18	-4.9480	-43.2252	213.3330
AT 19	-0.8859	4.3859	213.3330
AT 20	-10.8653	-23.6868	213.3330
AT 21	5.2039	34.9243	213.3330
AT 22	0.3065	6.2899	213.3330
AT 23	8.1343	98.8074	213.3330
AT 24	9.9812	119.8616	213.3330
AT 25	-20.9681	-213.3330	213.3330
AT 26	1.3990	-161.9017	213.3330
AT 27	-1.4389	-182.0982	213.3330
AT 28	-21.5518	-209.6402	213.3330
AT 29	-12.3581	-136.5671	213.3330
AT 30	-13.9473	-151.2701	213.3330
AT 31	-9.9408	-122.2506	213.3330
AT 32	-15.1231	-178.6258	213.3330



DEFORMATION IDENTIFICATION SEQUENCE

HEYMAN'S FRAME



PLASTIC HINGE NO. 11 FORMED AT POINT 7

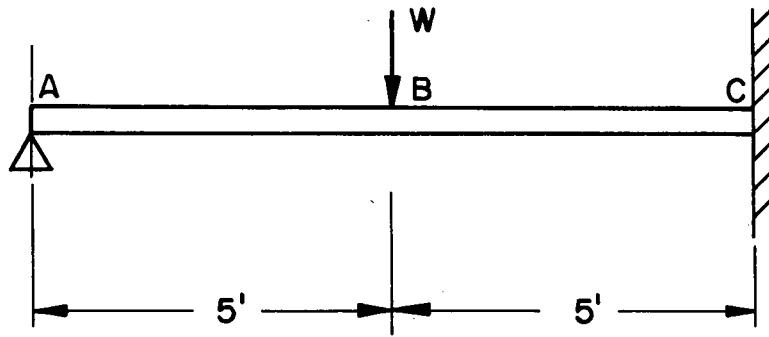
LOAD FACTOR	ADDITIONAL	CUMULATIVE	
STAGE 11	0.2017249E-01	0.2233189E 01	
DEFORMATION	ADDITIONAL	CUMULATIVE	
AT 1	0.7338172E-01	0.2126737E 00	
AT 2	0.1432835E-01	0.9152327E-01	
AT 3	0.1047941E 00	0.2057828E 00	
AT 4	0.1386438E 00	0.3778240E 00	
AT 5	0.3018041E 00	0.9125587E 00	
AT 6	0.2580480E 00	0.8139153E 00	
AT 7	0.2038937E 00	0.6510125E 00	
AT 8	0.1071458E 00	0.3350324E 00	
AT 9	0.4647286E-02	0.1429244E-01	
AT 10	-0.4892073E-02	-0.8470476E-02	
AT 11	-0.4892077E-02	-0.1271767E-01	
AT 12	0.2404476E-02	0.9047118E-02	
AT 13	-0.2776098E-03	-0.1809422E-02	
AT 14	0.4079600E-02	0.6897758E-02	
AT 15	0.676485E-02	0.1908631E-01	
AT 16	-0.6986274E-02	-0.9450560E-02	
AT 17	0.4729477E-02	0.1119317E-01	
AT 18	0.9021518E-02	0.3055573E-01	
AT 19	0.9353626E-02	0.1610212E-01	
AT 20	0.8495571E-02	0.2331155E-01	
MOMENT	ADDITIONAL	CUMULATIVE	PLAS MOM
AT 1	-9.0792	-154.9372	318.3330
AT 2	0.0000	-318.3330	318.3330
AT 3	0.0032	318.3365	318.3330
AT 4	0.0031	213.3363	318.3330
AT 5	41.1791	-49.9351	318.3330
AT 6	-25.1280	-318.3329	318.3330
AT 7	25.1282	318.3330	318.3330
AT 8	0.0000	318.3330	318.3330
AT 9	-9.0772	45.0656	350.0000
AT 10	0.0000	-350.0000	350.0000
AT 11	0.0003	350.0002	350.0000
AT 12	0.0000	350.0000	350.0000
AT 13	-9.0776	45.0655	350.0000
AT 14	0.0001	-349.9998	350.0000
AT 15	0.0000	350.0000	350.0000
AT 16	0.0000	350.0000	350.0000
AT 17	0.0000	-213.3330	213.3330
AT 18	1.6609	-59.4314	213.3330
AT 19	7.4169	14.3666	213.3330
AT 20	-19.5373	-122.6293	213.3330
AT 21	28.6153	77.5649	213.3330
AT 22	-23.4674	-42.3477	213.3330
AT 23	-17.7110	92.2839	213.3330
AT 24	9.0782	154.9363	213.3330
AT 25	0.0000	-213.3330	213.3330
AT 26	-7.7623	-189.2230	213.3330
AT 27	7.7624	-160.7768	213.3330
AT 28	0.0000	-213.3330	213.3330
AT 29	0.0001	-136.6664	213.3330
AT 30	-7.7624	-187.9727	213.3330
AT 31	7.7626	-130.3601	213.3330
AT 32	0.0000	-213.3330	213.3330

DEFLECTION TOO LARGE

COLLAPSE MECHANISM HAS BEEN REACHED

HINGE ROTATIONS

AT POINT 1	-0.3000000E-09
AT POINT 2	0.1615354E-01
AT POINT 3	0.9000000E-09
AT POINT 4	-0.3000000E-09
AT POINT 5	0.0000000E 00
AT POINT 6	0.1400000E-08
AT POINT 7	-0.1700000E-08
AT POINT 8	-0.8707182E-02
AT POINT 9	-0.1900000E-08
AT POINT 10	0.1408325E-01
AT POINT 11	0.4000000E-09
AT POINT 12	-0.2064373E-01
AT POINT 13	-0.1500000E-08
AT POINT 14	0.0000000E 00
AT POINT 15	-0.3702211E-01
AT POINT 16	-0.4423134E-01
AT POINT 17	0.1767095E-01
AT POINT 18	-0.1100000E-08
AT POINT 19	0.2600000E-08
AT POINT 20	0.2800000E-08
AT POINT 21	0.1600000E-08
AT POINT 22	0.1000000E-08
AT POINT 23	0.2700000E-08
AT POINT 24	0.1300000E-08
AT POINT 25	0.2129304E-01
AT POINT 26	0.1800000E-08
AT POINT 27	0.4900000E-08
AT POINT 28	0.7718326E-02
AT POINT 29	0.2700000E-08
AT POINT 30	0.2400000E-08
AT POINT 31	0.4500000E-08
AT POINT 32	0.1266886E-01



(a) Assume  $EI = 5000$  kip.ft.  
 $M_p = 40$  kip.ft.

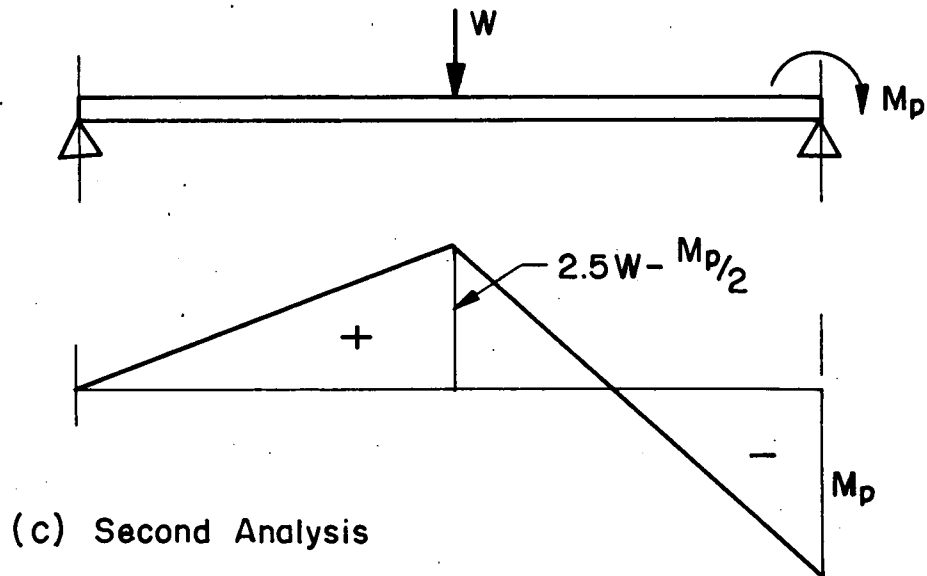
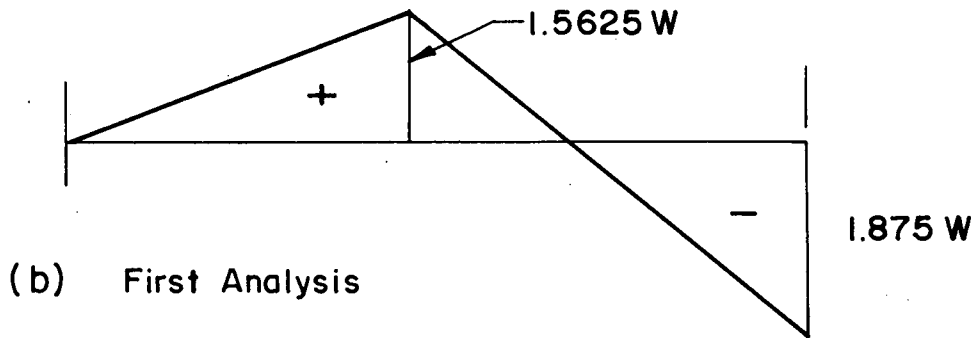


Fig. 1 Elastic-Plastic Analysis (manual computation)

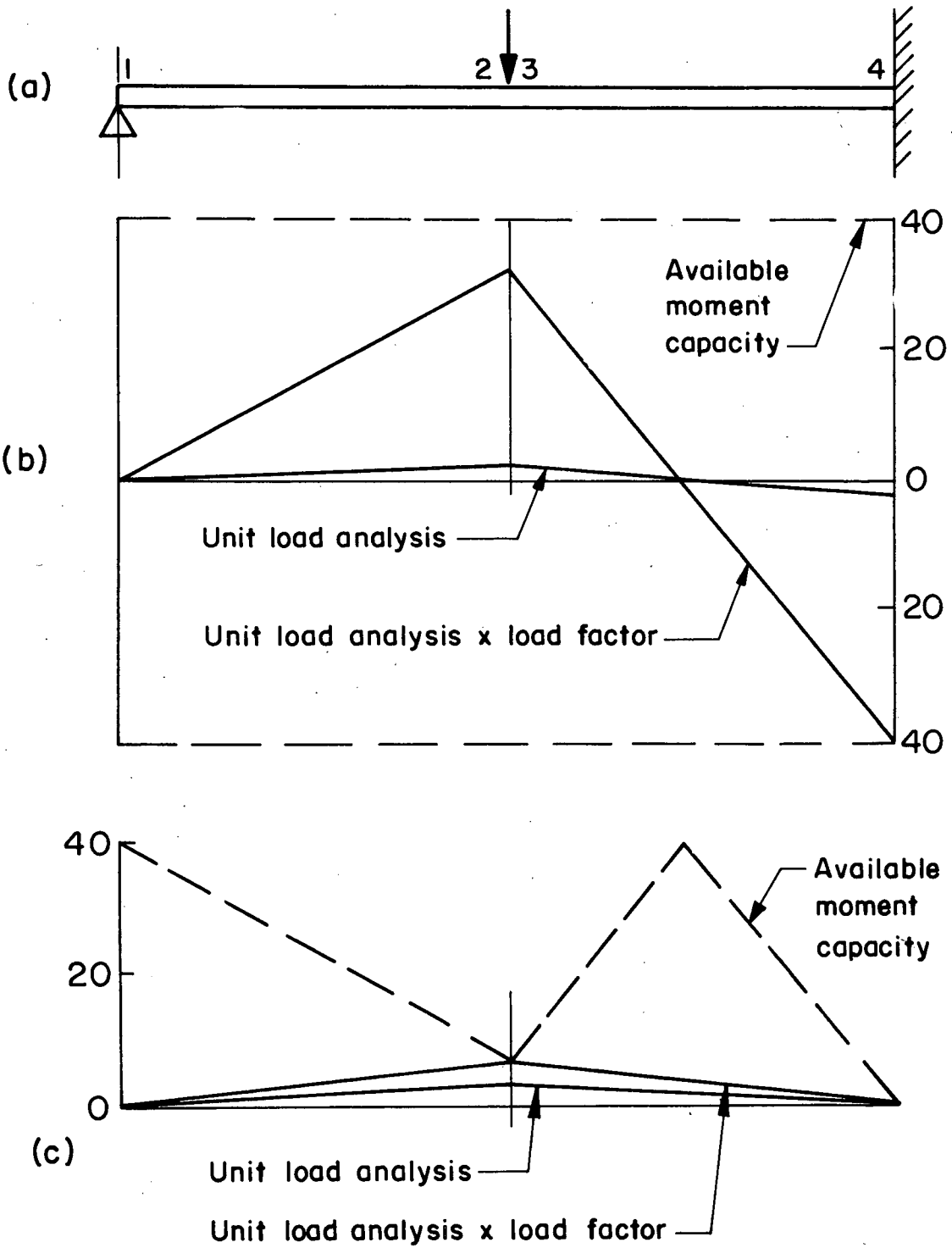


Fig. 2 Elastic-Plastic Analysis (machine computation)

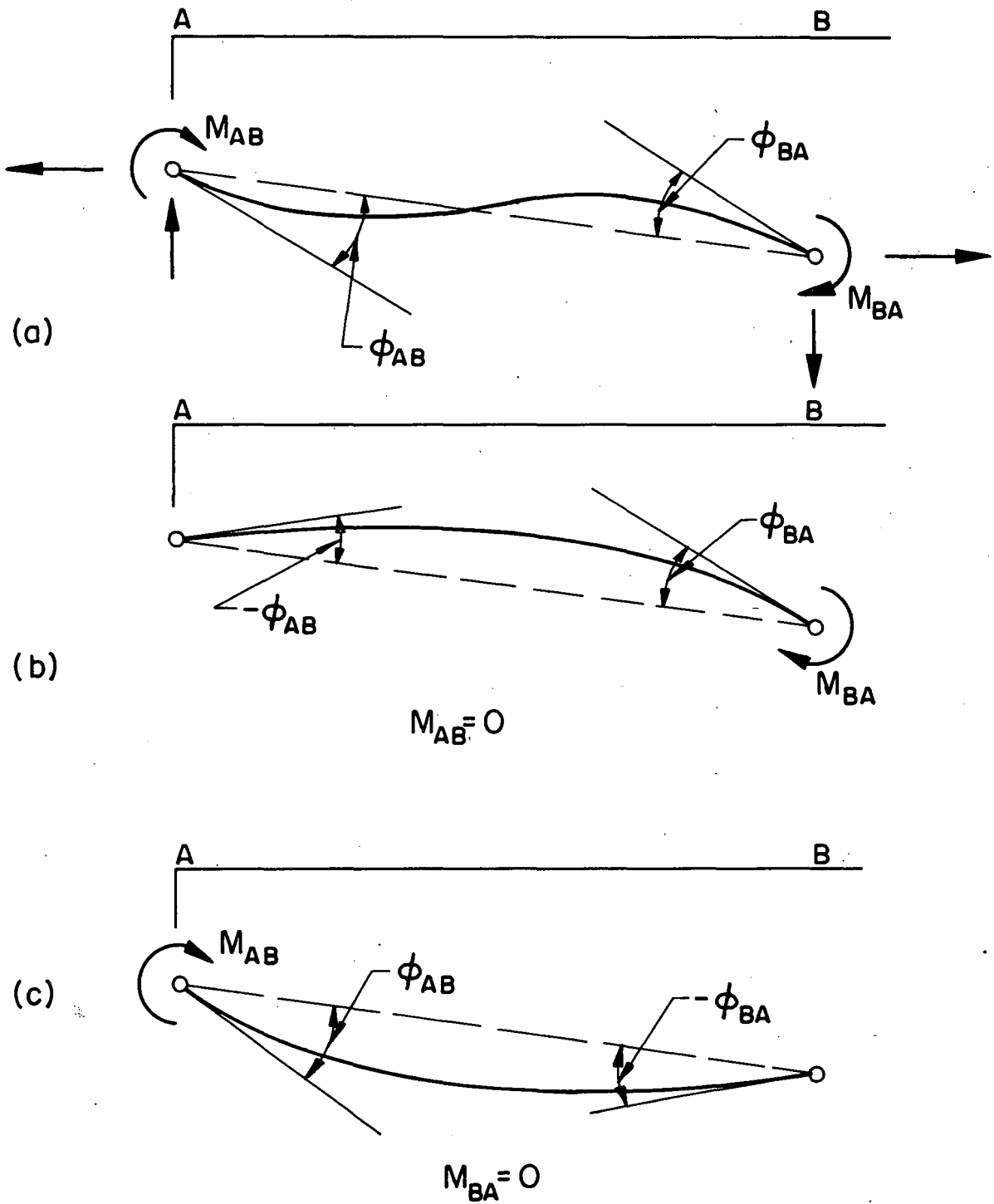


Fig. 3 Load Displacement Relations

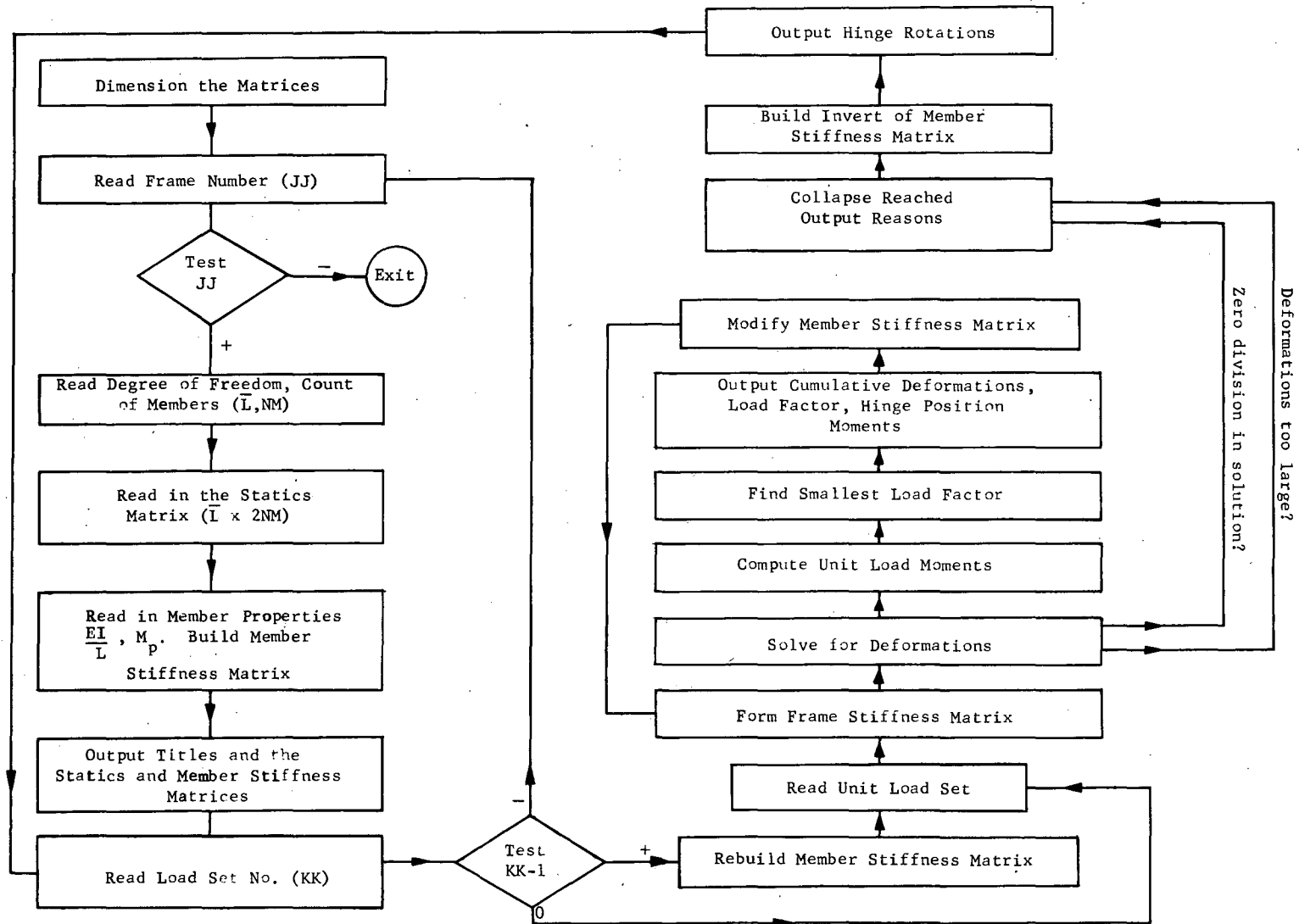
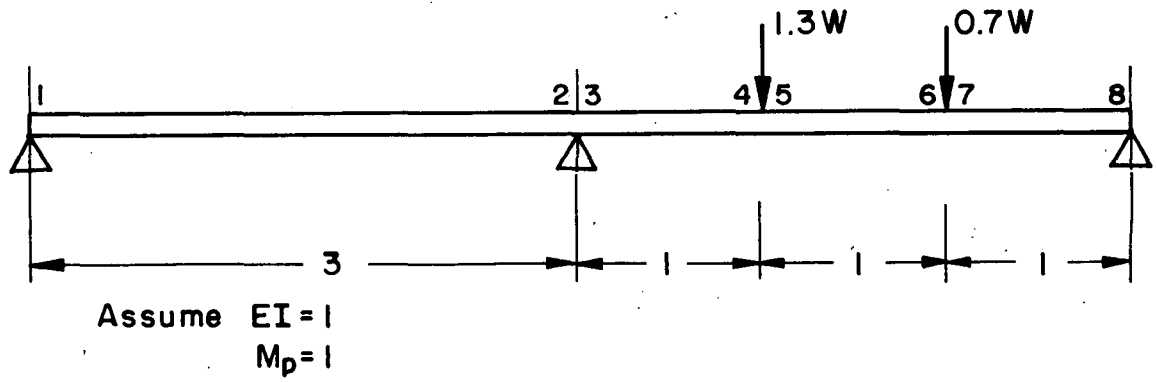
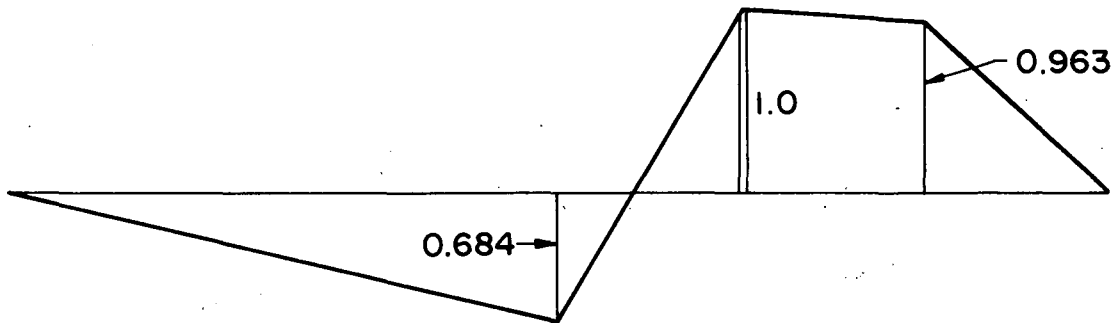


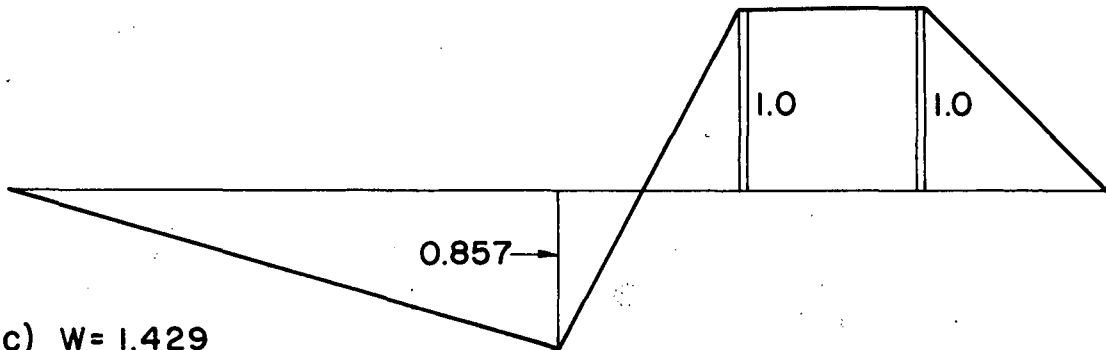
Fig. 4 Flow Diagram



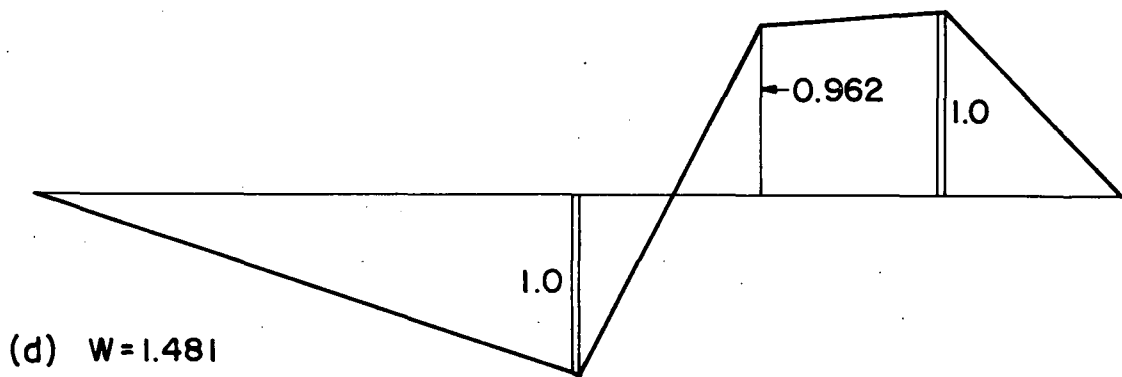
(a)



(b)  $W = 1.324$

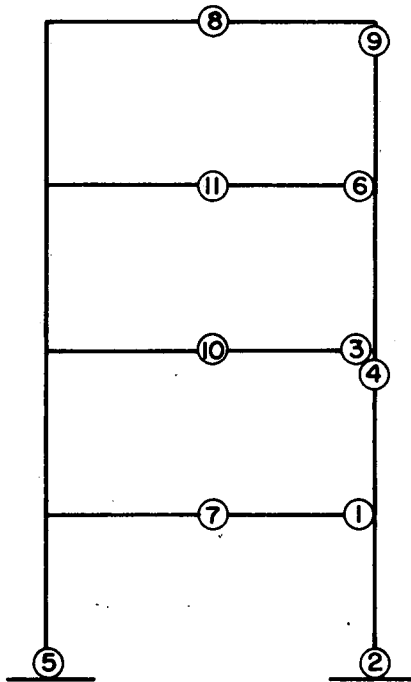
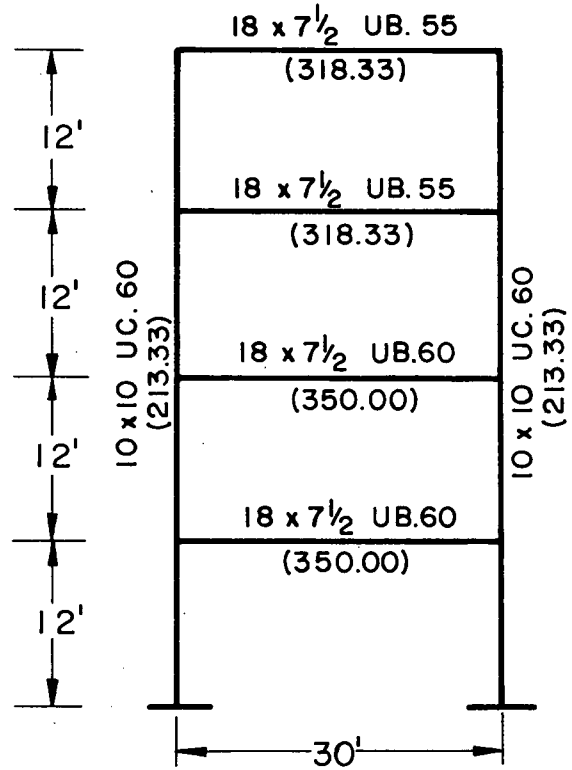
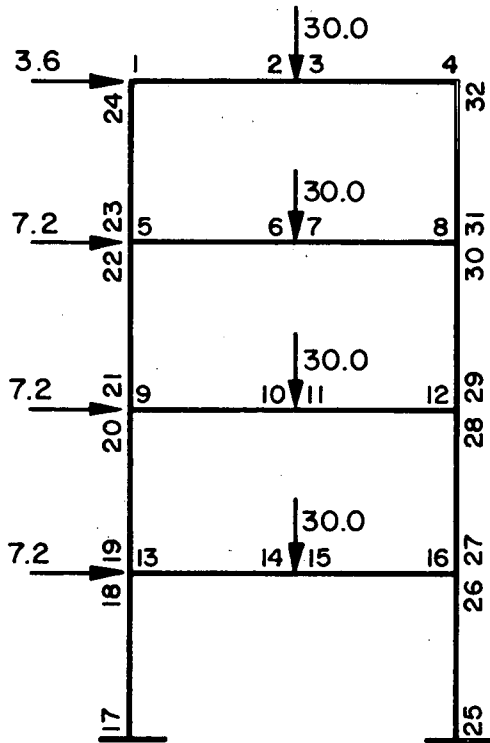


(c)  $W = 1.429$



(d)  $W = 1.481$

Fig. 5 Neal-Finzi Problem



UB - Universal Beam  
 UC - Universal Column  
 Units - Kip, Feet

HINGE No.	AT POINT	LOAD FACTOR	TOP SWAY
1	16	1.742	0.229
2	25	1.903	0.268
3	12	1.920	0.273
4	28	1.950	0.287
5	17	2.145	0.375
6	8	2.150	0.379
7	15	2.161	0.390
8	2	2.171	0.429
9	32	2.191	0.500
10	10	2.213	0.611
11	7	2.233	0.913

Fig. 6 Heyman's Frame

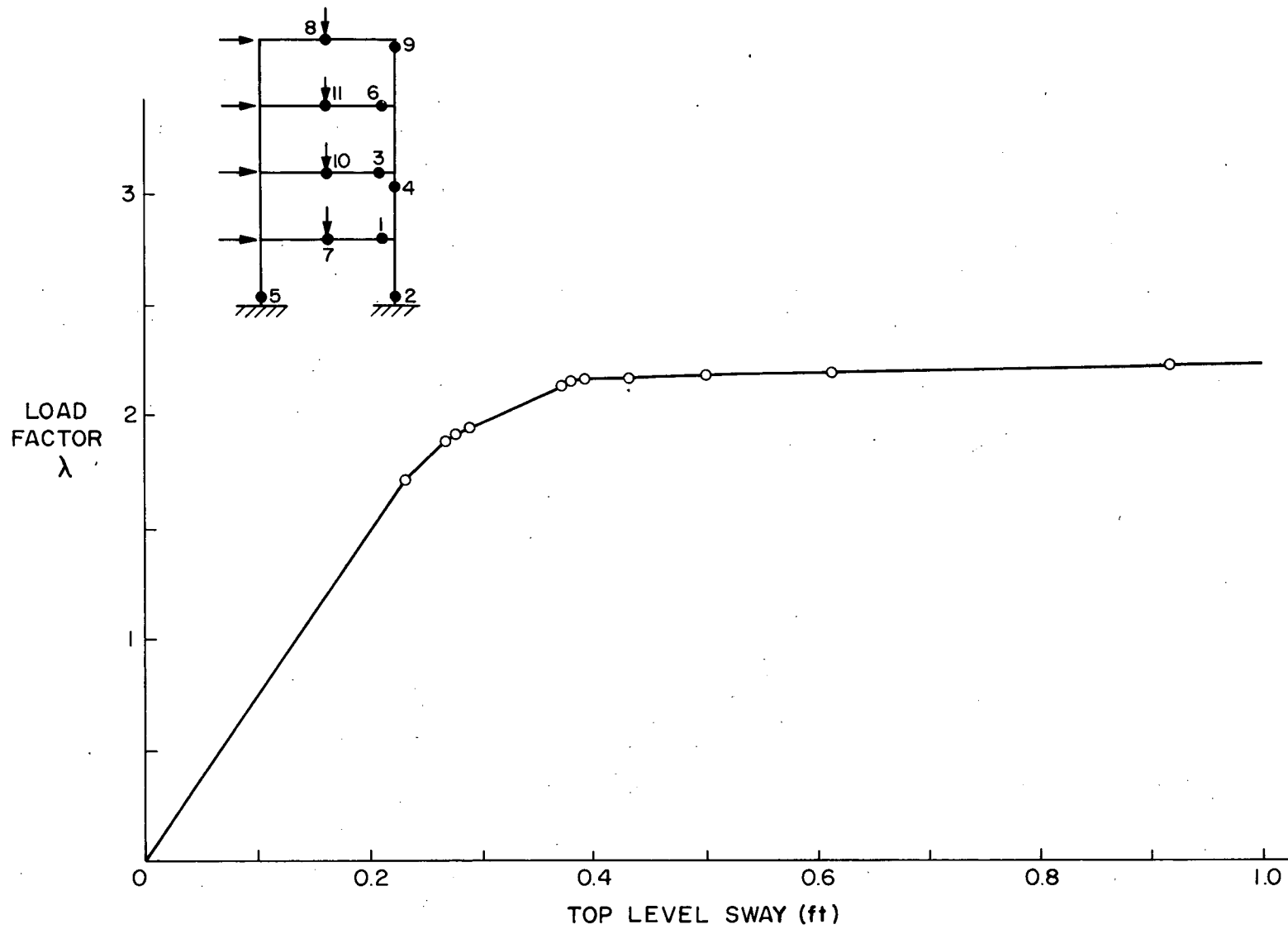


Fig. 7 Computed Load-Sway Curve



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