



Thermal Modeling of an Advanced Geared Turbofan for Distributed Engine Control Application

Jonathan Kratz (NASA GRC)

Dennis Culley (NASA GRC)

George Thomas (N&R Engineering)

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Outline

Summary

- Presents a method for modeling the dynamic thermal environment of a gas turbine engine with an eye toward control system component reliability as it relates to the implementation of distributed engine control.
- Application is to a conceptual N+3 generation geared turbofan.
- The resulting model is shown to run in real-time within a multi-model simulation environment that demonstrates the ability to interact with hardware to drive test equipment

Outline

- Background/Motivation
- Thermal Modeling Methodology
- Thermal Modeling Techniques
- Application to an Advanced Geared Turbofan
- Real-Time Capabilities
- Summary



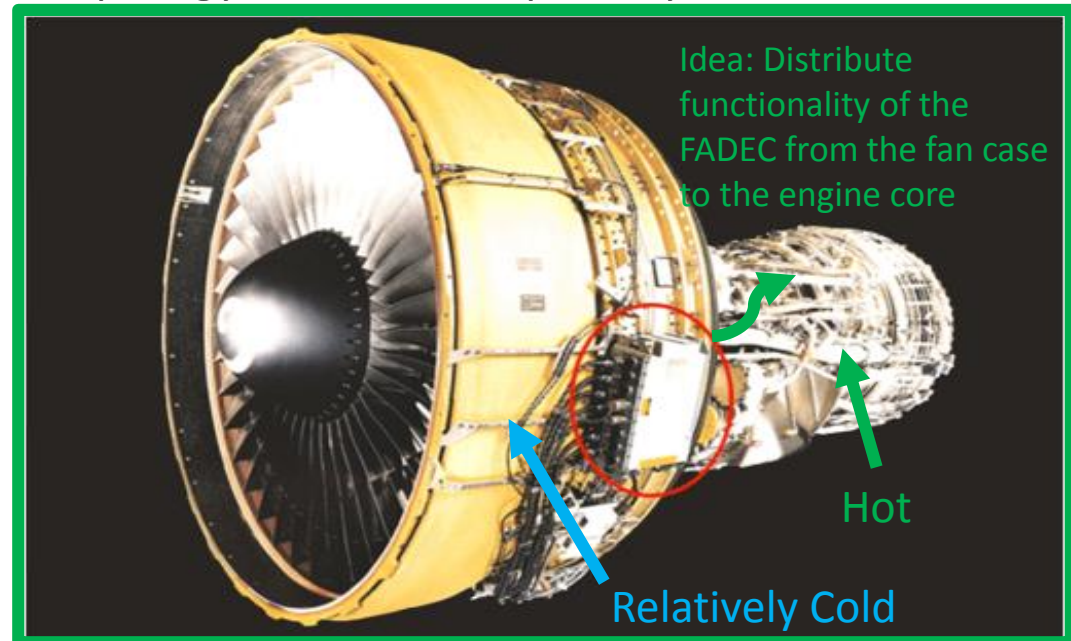
Background: Distributed Engine Control

Current Control Approach

- Centralized architecture performed through a full authority digital engine controller (FADEC)
- Constrains the control system topology and limits capability

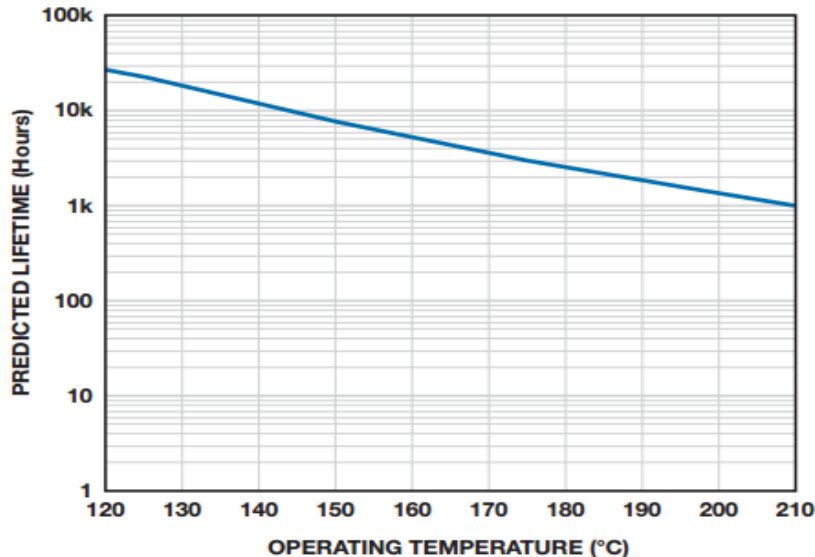
Distributed Engine Control

- Hardware-driven strategy that adds flexibility
- Modularizes the control system and distributes control functions to smart nodes located across the engine
- Utilizes a light-weight digital communication network
- Some Potential Benefits: Reduce weight, reduce volume impact, alleviate obsolescence and certification issues, enable more advanced control





Background: High Temperature Electronics



Example of the reliability vs. temperature relationship for an electronic device http://www.analog.com/library/analogdialogue/archives/46-04/high_temp_electronics.pdf

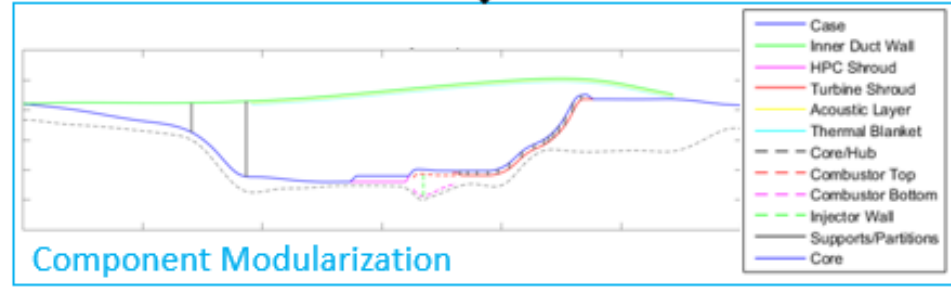
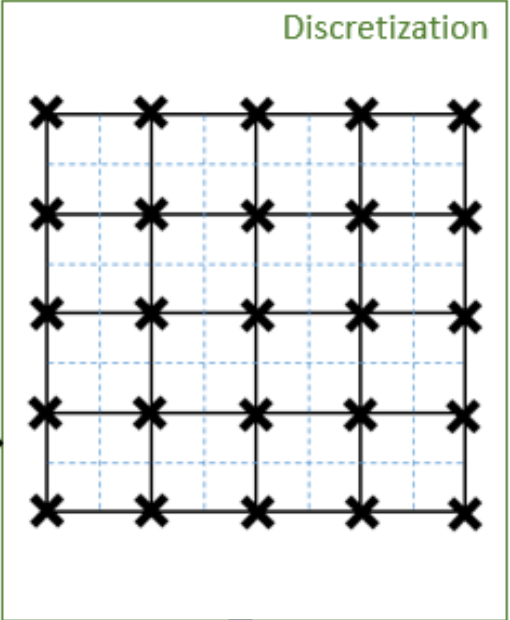
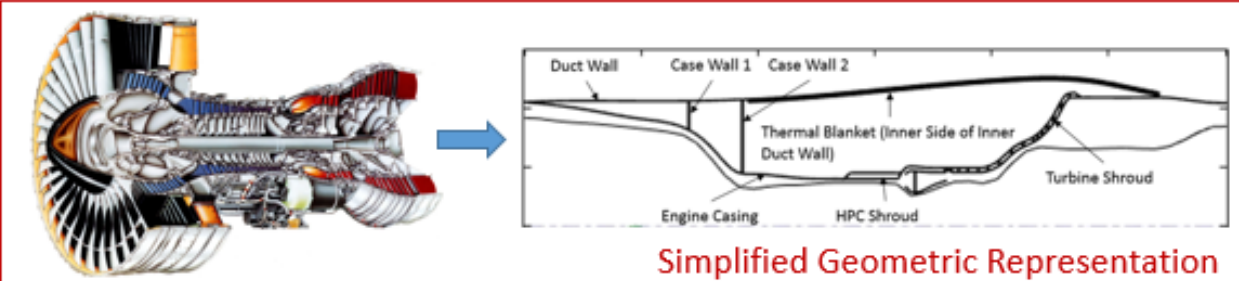
- Desire to mount smart nodes on the engine core
- Challenging thermal environment
 - State-of-art material for internal gas path exceed 1500°C
 - Common consumer electronics operate reliably <70°C, sometimes 150°C
 - Inverse relationship between temperature and electronic reliability
- High-Temp electronics
 - Silicon-On-Insulator (SOI): Up to 300°C (225°C near term)
 - Silicon Carbide: 500°C +

- Important considerations: Max & min temperature (steady-state), rate of change in temperature (dynamic), & temperature cycling (dynamic)
- Objective: Develop a thermal model of the relevant engine structure to estimate the environment in which DEC electronics will be placed + develop re-useable modeling tools + develop capability to use the model and or its results to drive test equipment

“High temperature is relative”

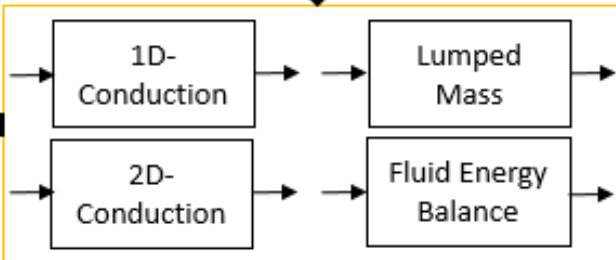
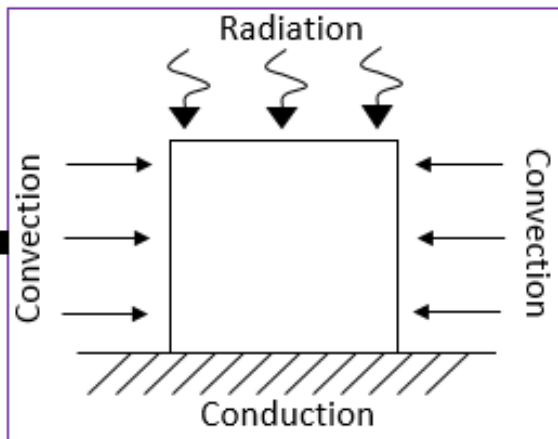
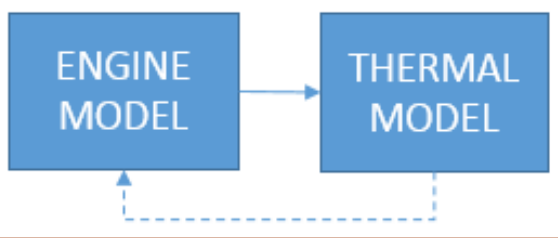


Thermal Modeling Methodology



Boundary Condition & Interface Definitions

Model Integration





Thermal Modeling Techniques

Structure

- 2-D Finite Difference Method (FDM) - More significant components (engine casing, shrouds, duct wall, etc.)

$$\frac{\partial T}{\partial t} = \frac{1}{\rho C_p} \nabla \cdot (k \nabla T) \rightarrow \frac{\partial T}{\partial t} = \frac{1}{\rho C_p} \left[\left(\frac{\partial k}{\partial r} + \frac{k}{r} \right) \frac{\partial T}{\partial r} + k \frac{\partial^2 T}{\partial r^2} + \frac{\partial k}{\partial z} \frac{\partial T}{\partial z} + k \frac{\partial^2 T}{\partial z^2} \right]$$

*T = temperature, t = time, ρ = density, C_p = heat capacity, k = thermal conductivity,
r = radial direction coordinate, z = axial direction coordinate*

- Model the component as a cylindrical shell of constant radius and thickness
- Discretized and then solved using a 2-D implicit scheme
- Lumped Capacitance – Less significant components (core components – compressor and turbine blades)

$$\frac{\partial T}{\partial t} = \frac{1}{m C_p} \left[\underbrace{h A (T_F - T)}_{\text{Convection}} + \underbrace{u A (T_R - T)}_{\text{Radiation}} \right]$$

*m = effective mass, A = surface area, T_F = temperature of convecting fluid, T_R = temperature of radiating body,
h = convection heat transfer coefficient, u = radiation heat transfer coefficient*



Thermal Modeling Techniques

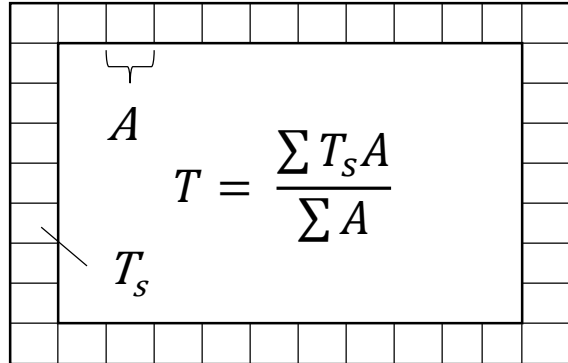
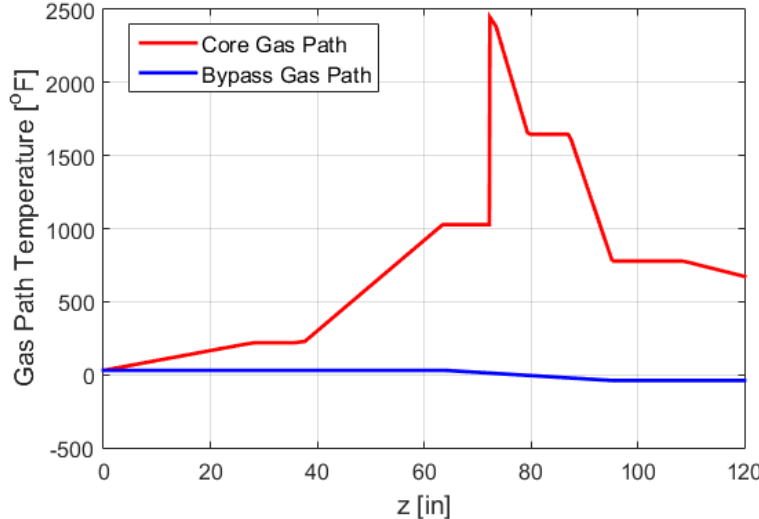
Flow Paths & Voids

- Engine simulation data – used for gas paths and some bleed flows
- Fluid Energy Balance – used for bleed flows of significant heat transfer and relatively low mass flow

$$Q = \sum_{out} (\dot{m}C_pT) - \sum_{in} (\dot{m}C_pT)$$

T = temperature, *C_p* = heat capacity,
ṁ = mass flow rate, *Q* = heat

- Average of Surroundings – used for closed volumes with no forced air flow



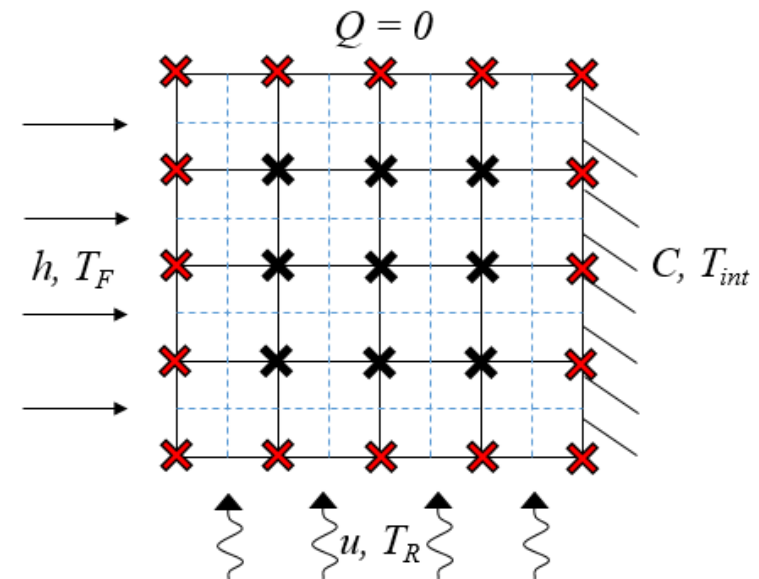
A = surface area of element exposed to the enclosed fluid
T_s = surface temperature of the element



Thermal Modeling Techniques

Boundary Conditions

- Conduction
 - Thermal capacitance of each boundary node is computed to enable conduction boundary conditions to be applied at the interface of 2 solid components
- Convection
 - Forced and natural convection are considered
 - Relations used for the coefficient h are generic and tunable
 - Tuning variables were set based on guidance from studies conducted at NASA and information found in literature
- Radiation
 - Written in a linear form
 - The coefficient u is a strong function of temperature and is updated each time-step of the simulation.
 - Relations for u assumes radiation between reflective concentric cylinders
 - Assumed radiation only occurs between parallel surfaces



$Q = \text{heat}$

$h = \text{convection heat transfer coefficient}$

$T_F = \text{fluid temperature}$

$u = \text{radiation heat transfer coefficient}$

$T_R = \text{temperature of radiating body}$

$C = \text{thermal capacitance}$

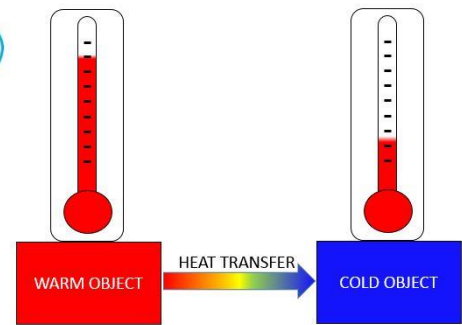
$T_{int} = \text{temperature at the interface between 2 solids}$



Thermal Modeling Techniques

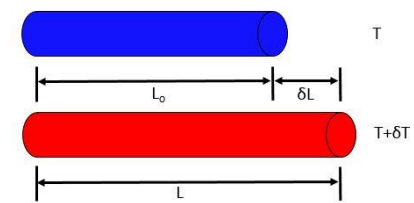
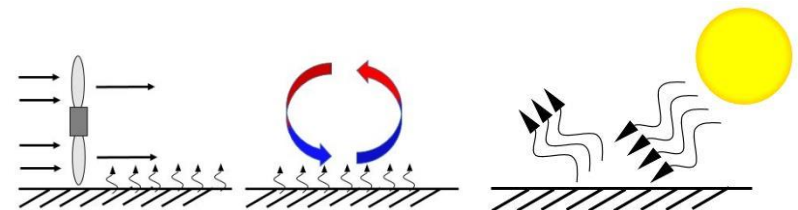
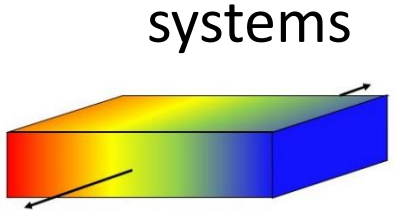
Thermal System Analysis Toolbox (TSAT)

- Library of tools developed in the MATLAB/Simulink environment
- Topics modeled
 - Conduction
 - Convection
 - Radiation
 - Deformation
 - Air Properties
 - Fluid Heat Transfer
 - General Tools
- Provides building blocks for building up and modeling dynamic thermal systems



> Thermal Conductivity Matrix [k]	
> Heat Capacity Matrix [Cp]	
> Density Matrix [rho]	
> Left Surface Convection BC [TfL, hL]	Temperature Array [T]
> Left Surface Radiation BC [TextL, uL]	
> Left Surface Conduction BC [TsL, CL]	
> Right Surface Convection BC [TfR, hR]	
> Right Surface Radiation BC [TextR, uR]	
> Right Surface Conduction BC [TsR, CR]	
> Bottom Surface Convection BC [TfB, hB]	
> Bottom Surface Radiation BC [TextB, uB]	
> Bottom Surface Conduction BC [TsB, CB]	Thermal Conductance [C]
> Top Surface Convection BC [TfT, hT]	
> Top Surface Radiation BC [TextT, uT]	
> Top Surface Conduction BC [TsT, CT]	

2D Transient Conduction Model - Fully Implicit



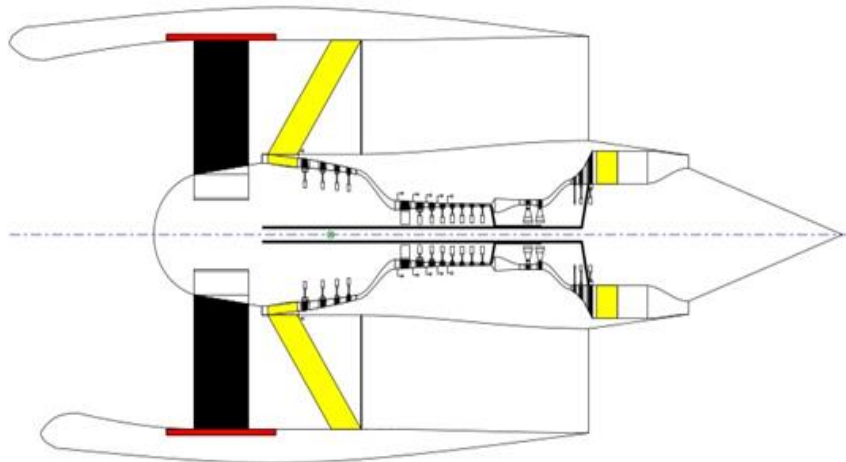
Download Link: <https://github.com/nasa/TSAT>



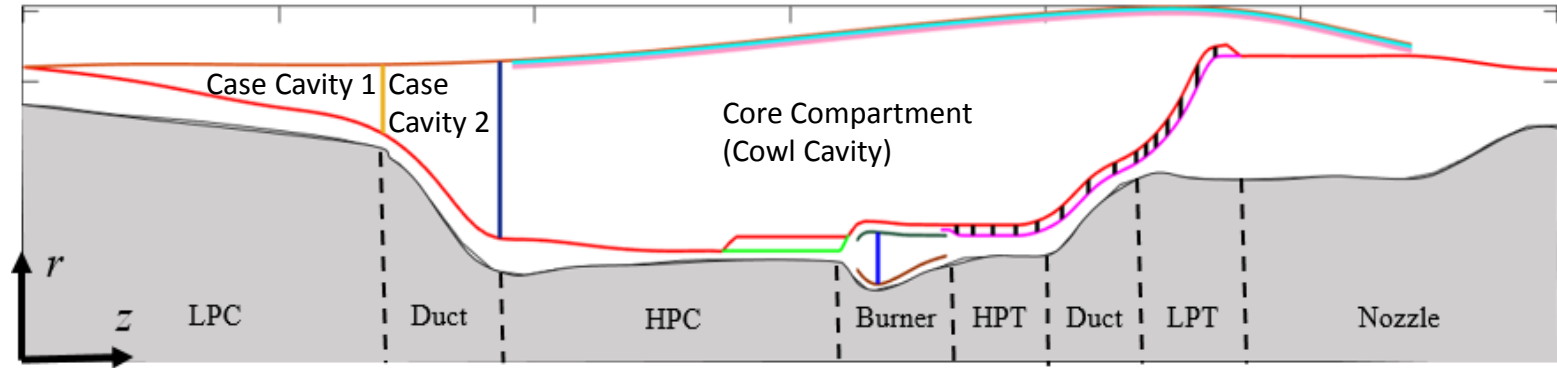
Application: The Engine

Advanced Geared Turbofan 30,000lb_f (AGTF30)

- Based on the NASA N+3 NPSS reference engine
- 3rd generation geared turbofan
- Features a compact gas turbine (CGT) and a variable area fan nozzle
- Capable of producing 30,000lb_f of thrust at the sea-level static condition

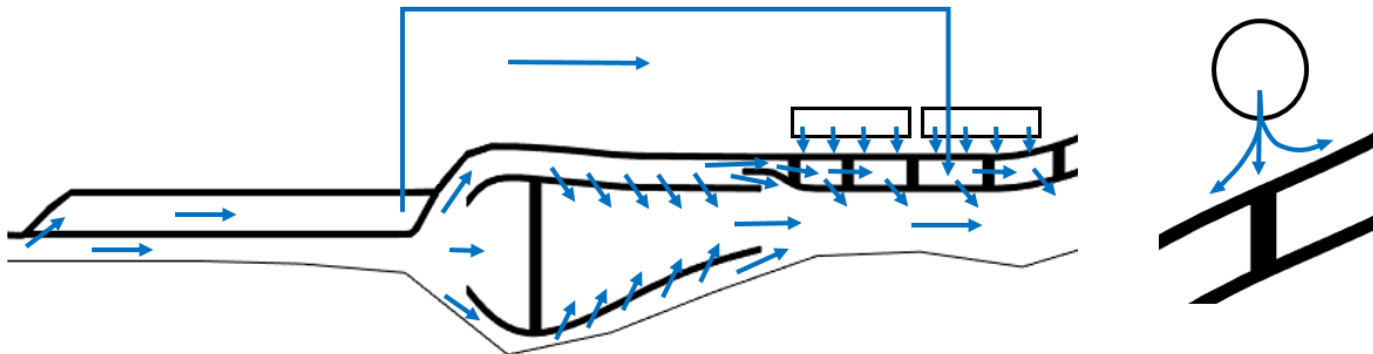
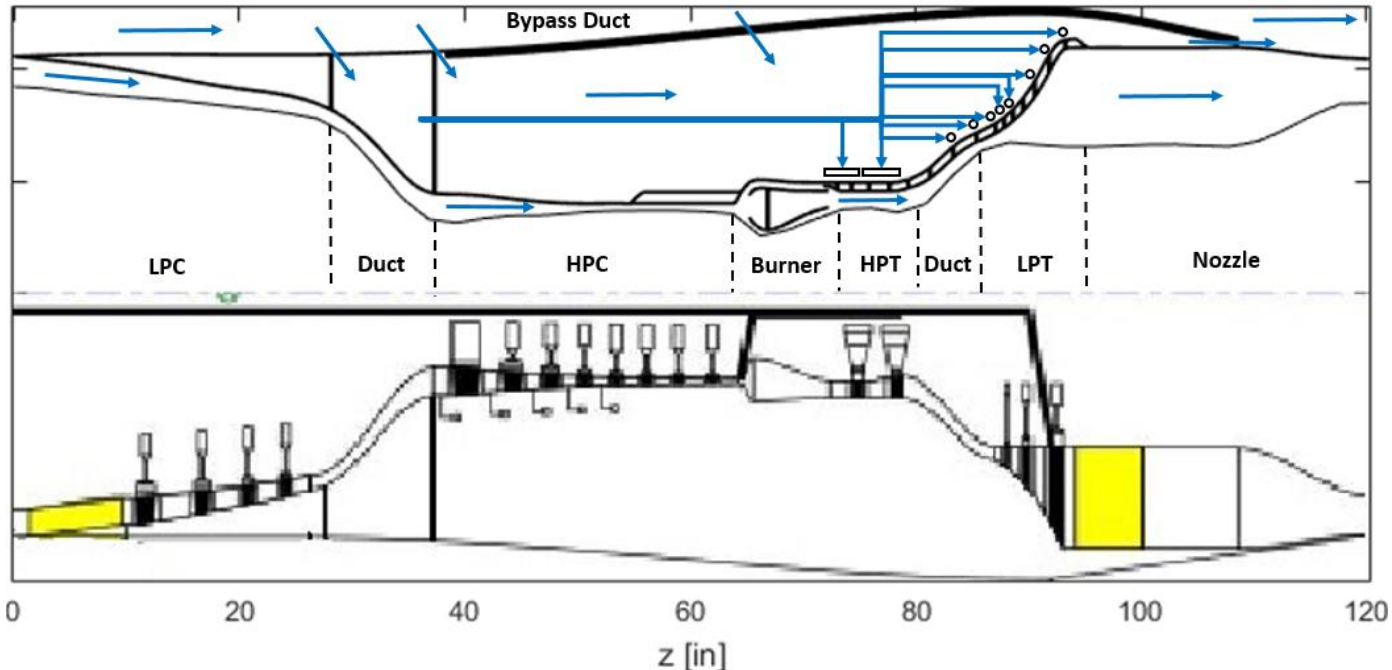


- | | | |
|------------------------------------|------------------------------|-------------------------------------|
| Engine Casing | HPT Shroud | Combustor partition – injector wall |
| Inner Duct Wall – Structural Layer | Case Wall 1 | Core Components |
| Inner Duct Wall – Acoustic Layer | Case Wall 2 | Shroud supports |
| Inner Duct Wall – Thermal Blanket | Combustor partition – top | |
| HPC Shroud | Combustor partition – bottom | |





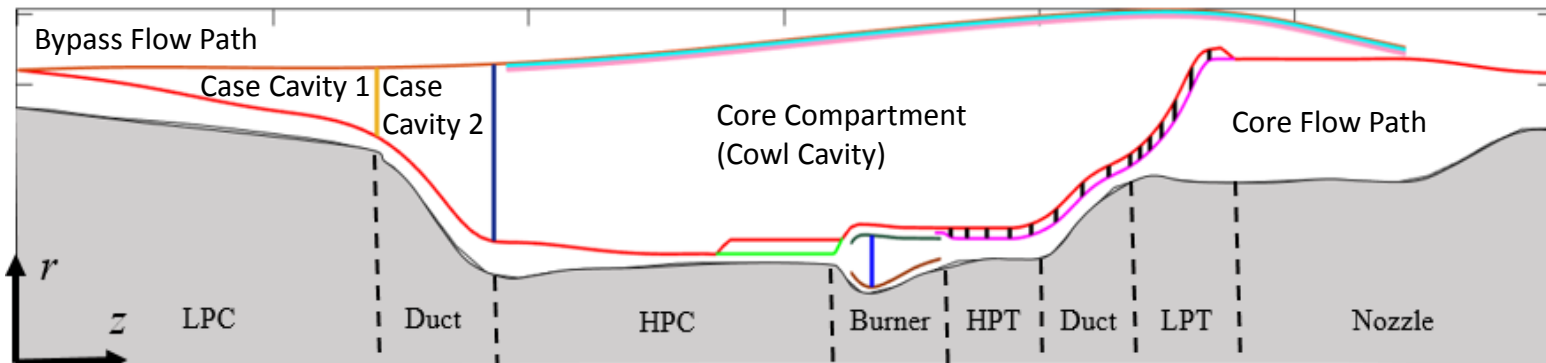
Application: The Engine Air Flow





Application: Modeling

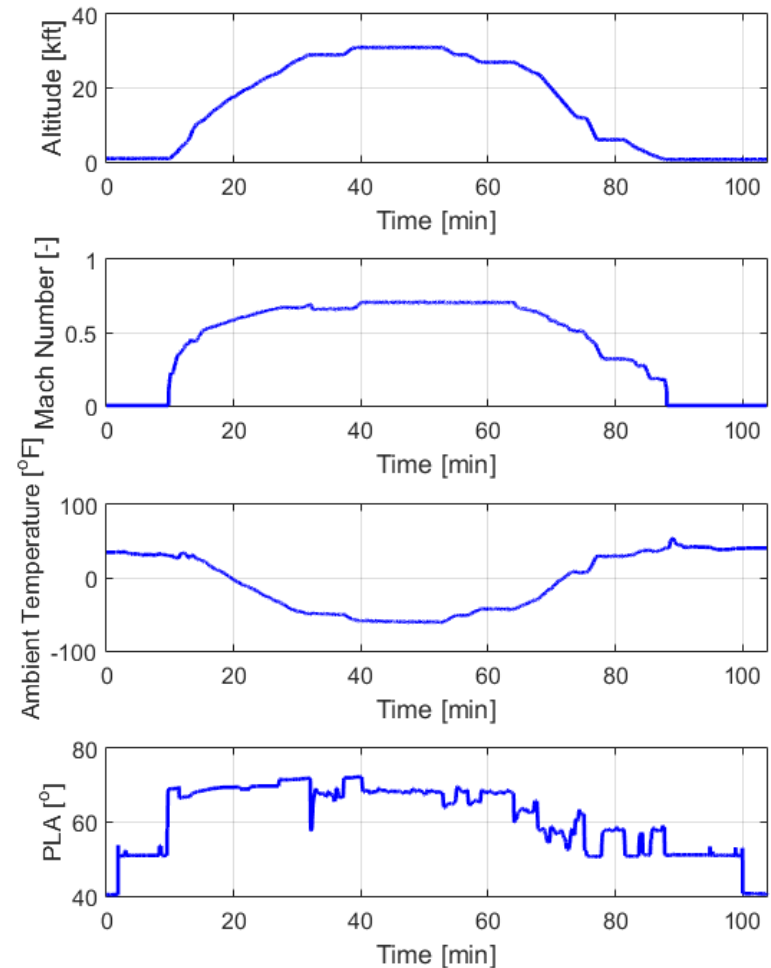
- Structures
 - All components except the “core components” were modeled with 2-D FDM
 - Geometry was approximated as a cylindrical shell of the components average radius and thickness
 - Various levels of discretization were investigated
 - Core components utilized a lumped capacitance model
- Important cavities and voids
 - Bypass flow path, core flow path, and case cavity 2 temperatures were driven by the engine model simulation
 - Case cavity 1 temperature was approximated as the average of its surrounding structure
 - Core compartment temperature was approximated using the fluid energy balance method
- Boundary conditions and interfaces between models were defined





Application: Flight Profile

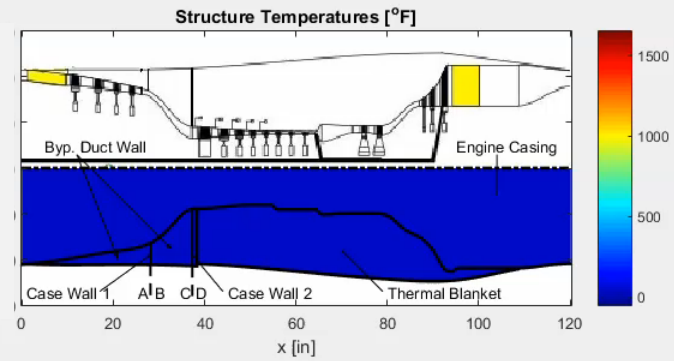
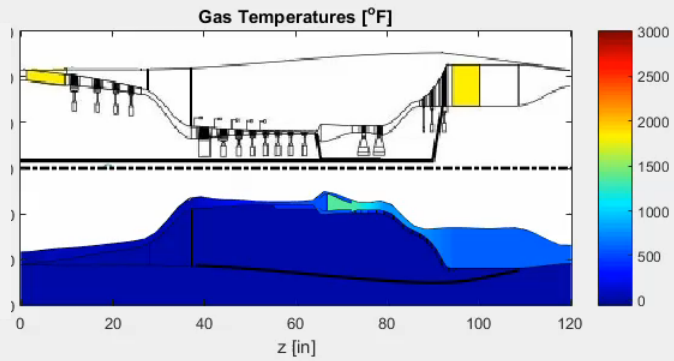
- Flight profile constructed from real data
- The starting and ending destinations are unknown but is representative of a ~250 mi flight
 - Cleveland, Ohio to Washington D.C. or Las Vegas, Nevada to Los Angeles, California
- At the start of the simulation all structures are initialized at ambient temperature
- After the flight, the thermal simulation is extended to investigate heat soak back (modeling details are not provided here for the sake of time – see the paper)



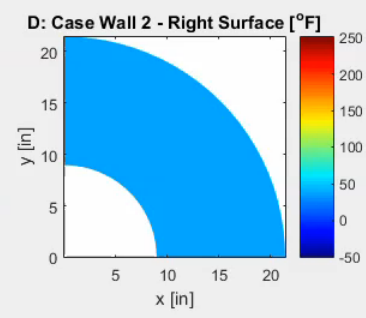
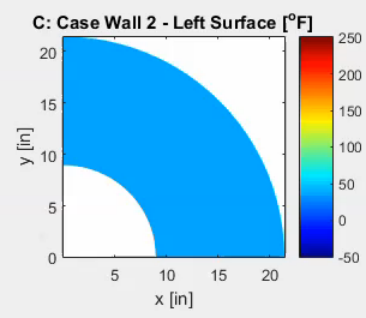
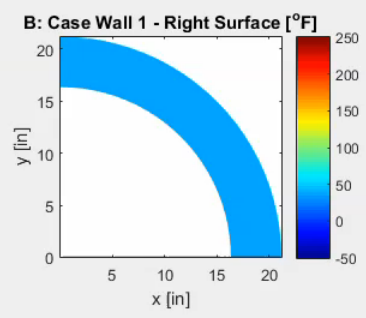
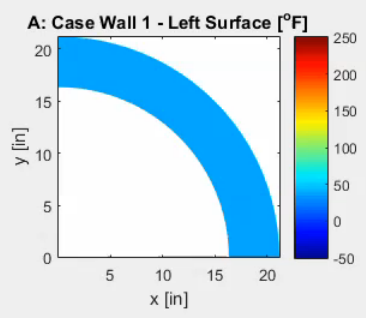
PLA = power lever angle



Application: Results



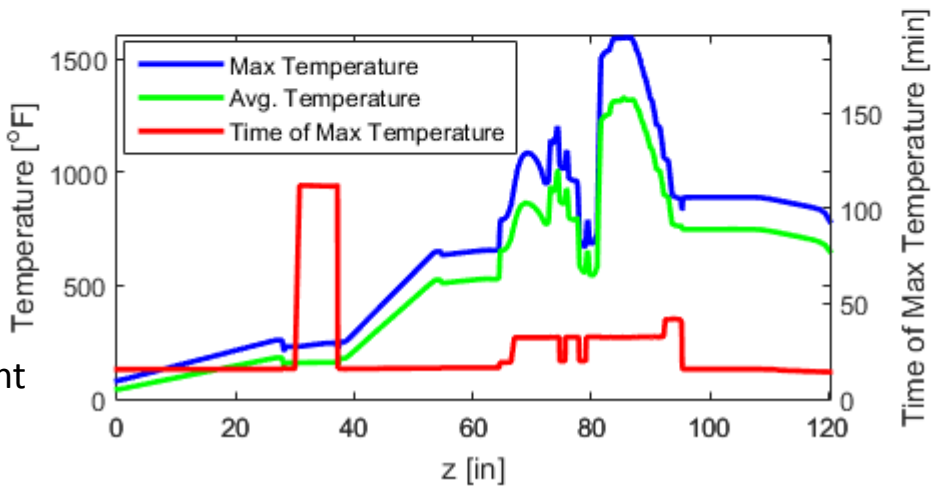
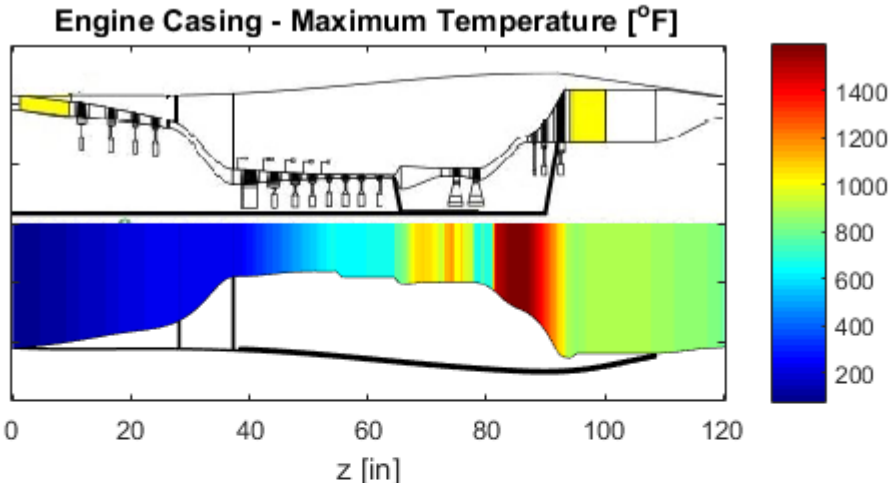
Time: 0min





Application: Results

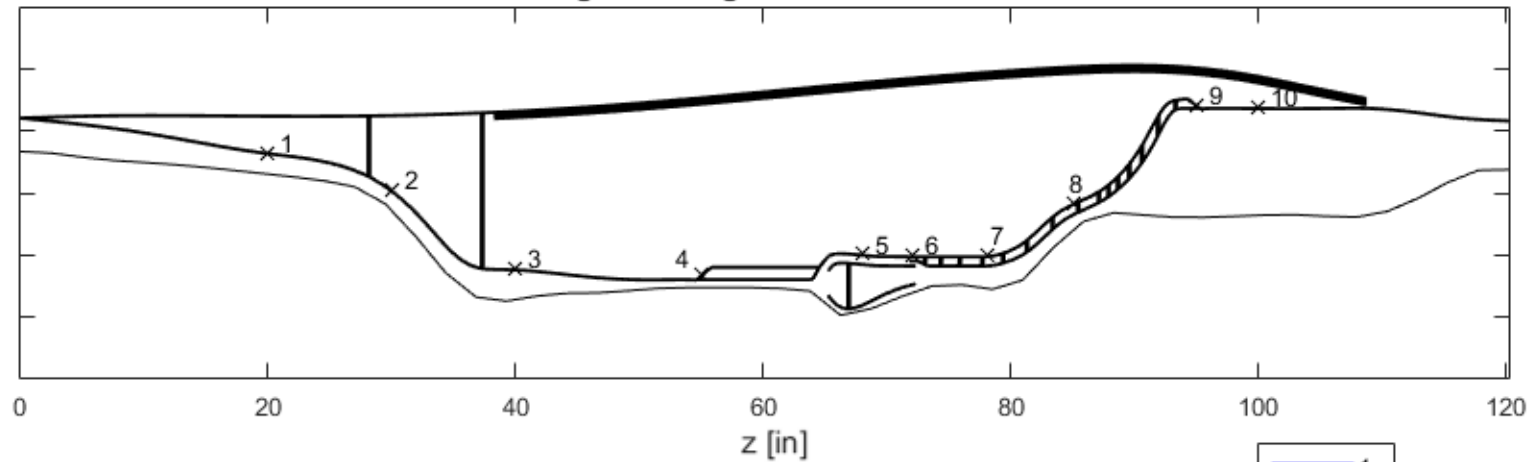
- Compartments
 - Case Cavity 1: Up to 125 °F
 - Case Cavity 2: Up to 155 °F
 - Core Compartment (Cowl Cavity): Up to 650 °F (400 °F upstream of the inter-turbine duct)
- Structure
 - Engine Casing: Up to 1580 °F (1100 °F outside the inter-turbine duct)
 - Inner Duct Wall (Thermal Blanket): Up to 600 °F (350 °F upstream of the inter-turbine duct)
 - Case Wall 1: Up to 220 °F
 - Case Wall 2: Up to 220 °F
- Observations
 - Rate of change in temperature
 - Compartments: -2.5 °F/sec – 2.5 °F/sec
 - Structures: -2.5 °F/sec - 5 °F/sec
 - Maximum temperatures occurred at different parts of the engine during different times including: the cold startup of the engine, during climb, during cruise, and during heat soak back



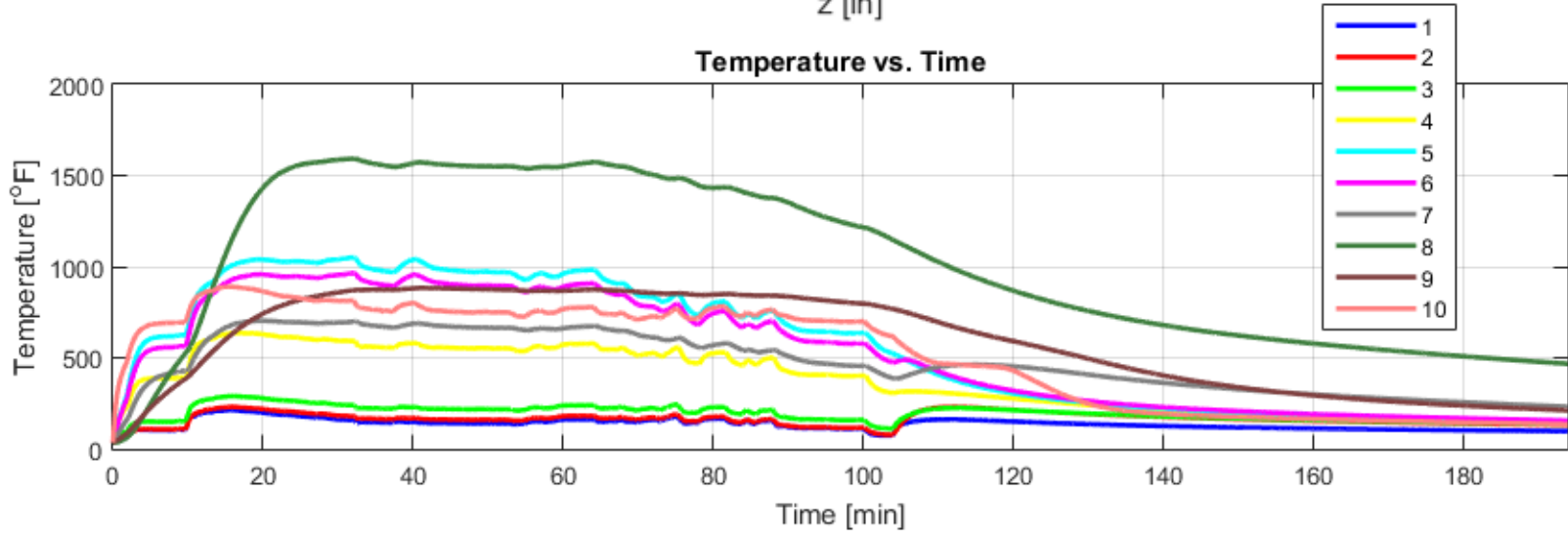


Application: Results

Engine Casing - Select Locations

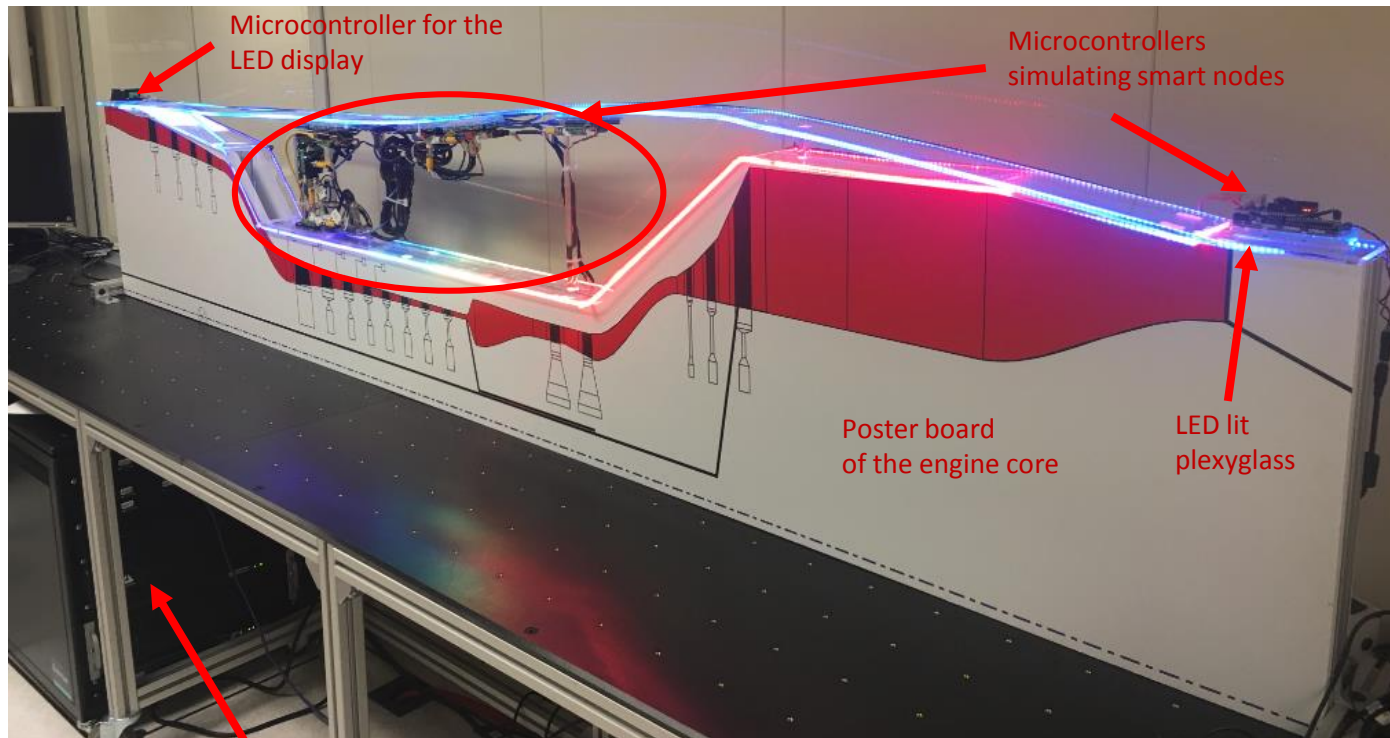


Temperature vs. Time



Real-Time Capabilities

- Model runs faster than real-time
- Migrated to the hardware-in-the-loop (HIL) system known as the Decentralized Engine Control System Simulator (DECSS)
- Integrated in a multi-model simulation including a physical network and simulated smart nodes
- Used the model to drive a real-time, full-size LED display
 - Illustrates the ability to interact with test equipment





Summary

- Motivation for high level thermal modeling pertaining to distributed engine control has been discussed
- A thermal modeling methodology for gas turbine engines has been proposed
- An application of modeling methodology has been illustrated
- Results from the application have been presented and discussed
- Real-time capabilities have been demonstrated with eye toward hardware testing



Acknowledgements

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- NASA civil servants & contractors who contributed in some way to this effort: Sanjay Garg, Scott Jones, Jonathan Litt, Jeffryes Chapman, Vikram Shyam, Paht Juangphanich, Ram Bhatt, Jerry Lang, James DiCarlo, Joe Grady, Dan Paxson, and Shane Sowers
- The Distributed Engine Control Working Group (DECWG[®]) for providing input and guidance related to this work



Questions?

Contact Information:

- Jonathan Kratz – jonathan.kratz@nasa.gov
- Dennis Culley – dennis.e.culley@nasa.gov
- George Thomas – george.l.thomas@nasa.gov

TSAT Link:

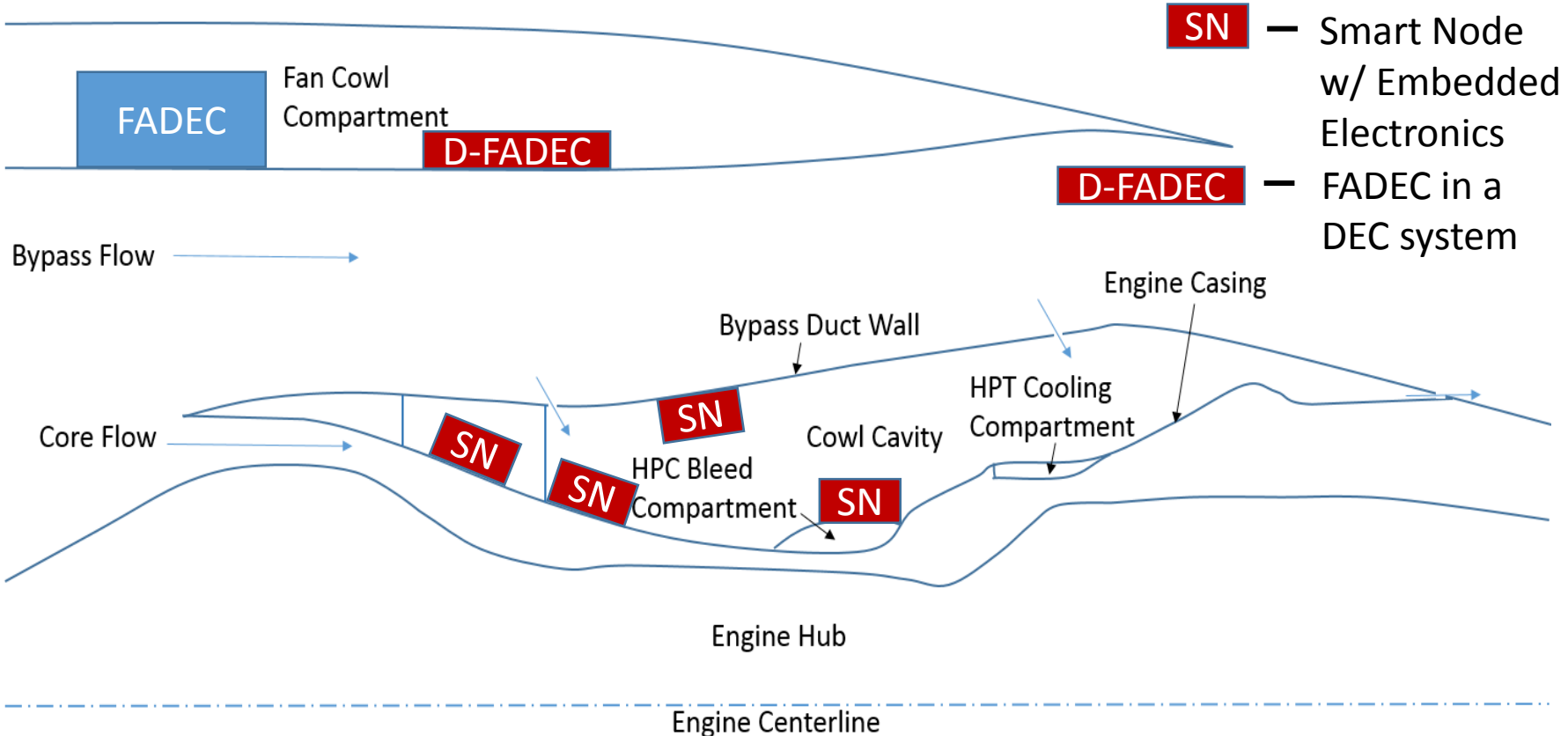
<https://github.com/nasa/TSAT>



EXTRA SLIDES



Background: Distributed Engine Control

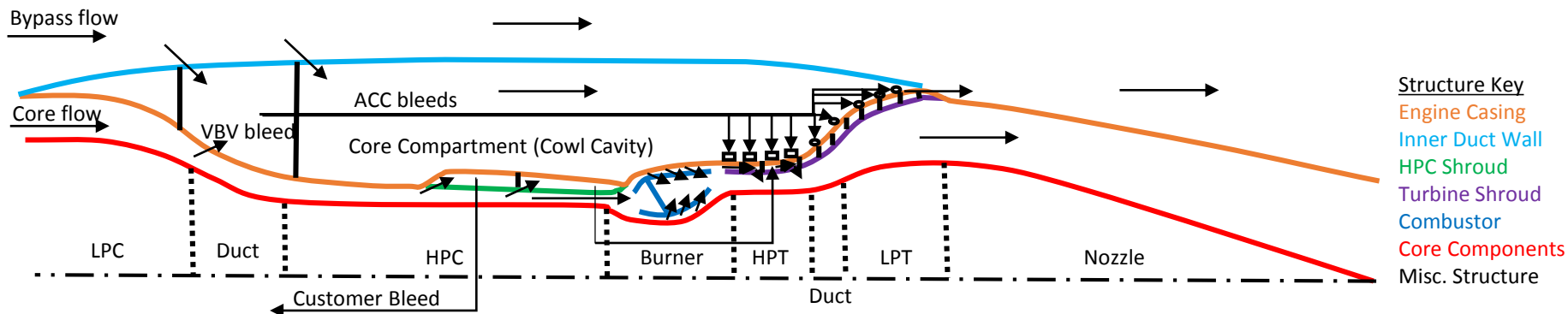


Where can we mount hardware?

- SN mounting surfaces could include those exposed to the cowl cavity or case compartments (engine casing, bypass duct wall, & various supports structures)



Thermal Modeling Methodology



What part(s) of the engine are we interested in modeling?

- Any potential mounting structure for a smart node
- Any compartment in which a smart node could be mounted
- Any structure or flow path that could have significant implications on the thermal environment of a potential mounting locations

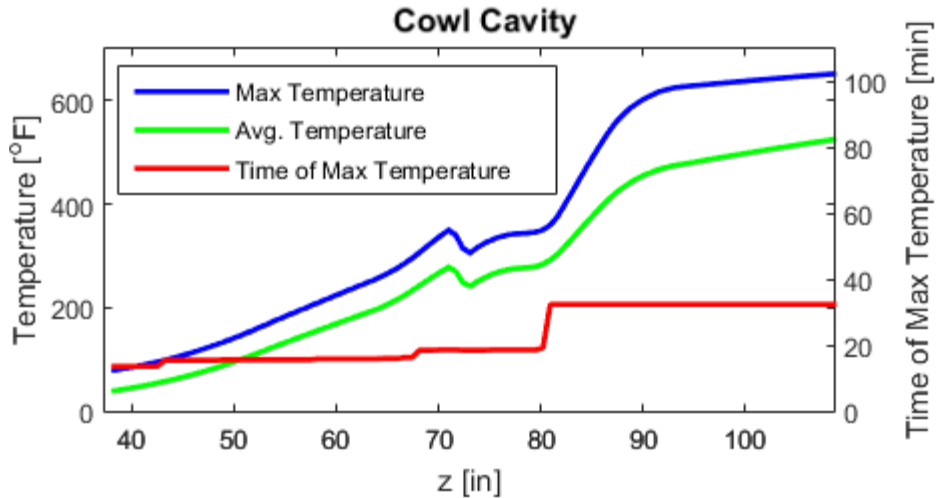
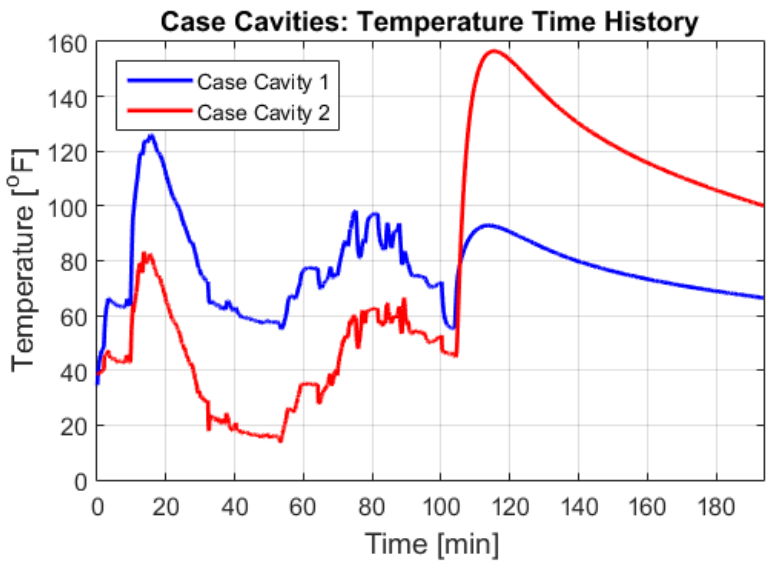
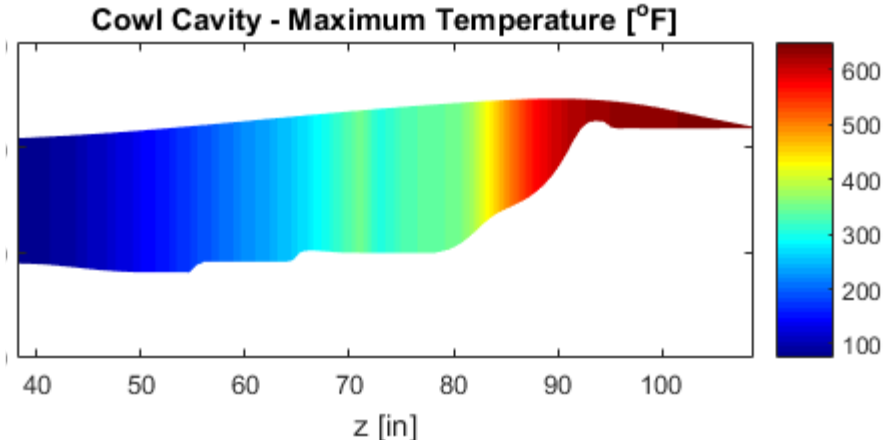
Needs?

- Geometry/dimensions, secondary air system, other heat transfer/cooling mechanisms inherent in the design



Application: Results

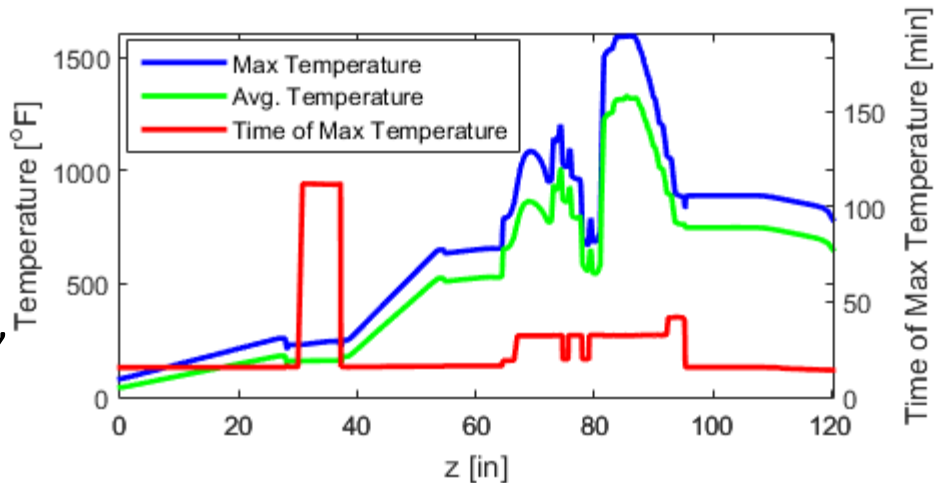
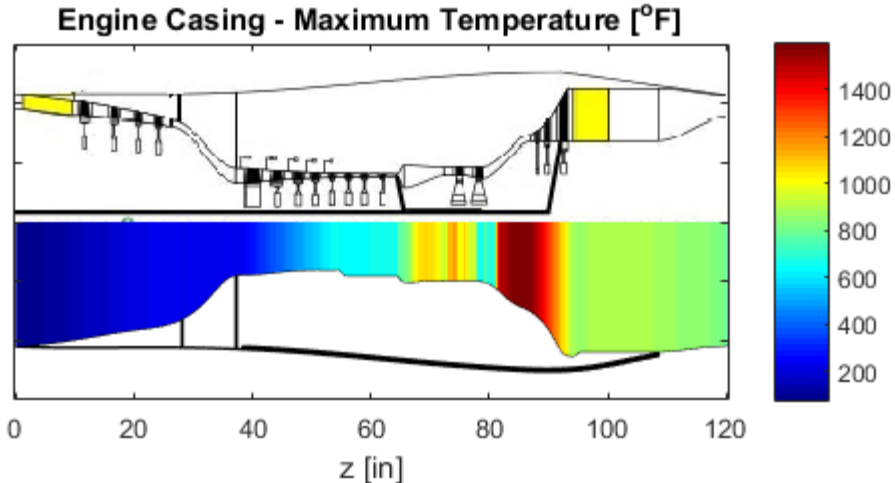
- Max T: 125°F during take-off (case cavity 1), 155 °F during heat soak (case cavity 2), 400 °F during climb (applicable portion of the cowl cavity)
- Max Increasing dT/dt: 1 °F/sec (case cavity 1 & 2), 2.5 °F/sec (applicable portion of the cowl cavity)
- Max Decreasing dT/dt: 0.5 °F/sec (case cavity 1), 2 °F/sec (case cavity 2), 2.5 °F/sec (applicable portion of the cowl cavity)
- Observations
 - Temperature rises several hundred degrees through the cowl cavity (core compartment)
 - Case cavity 1 shows dampening effects compared to the temperature response of case cavity 2





Application: Results

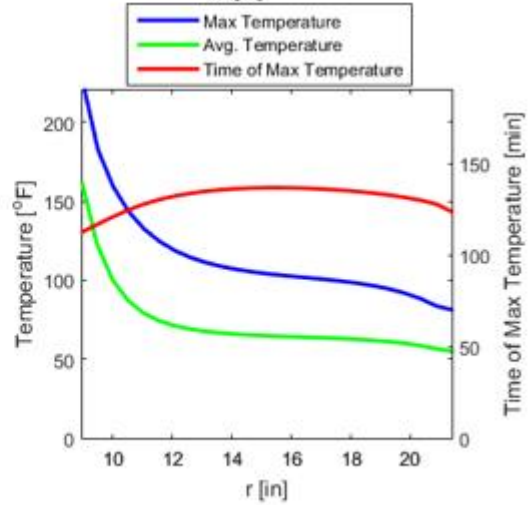
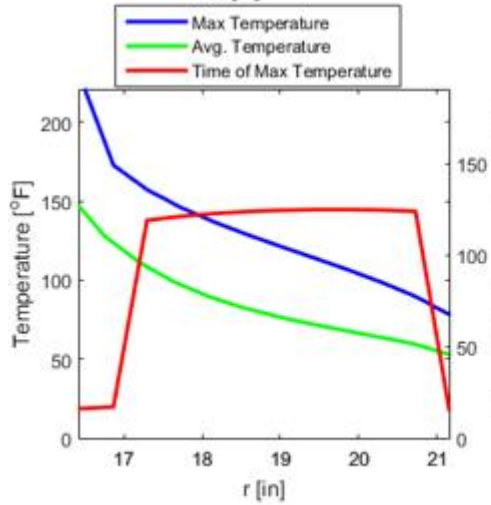
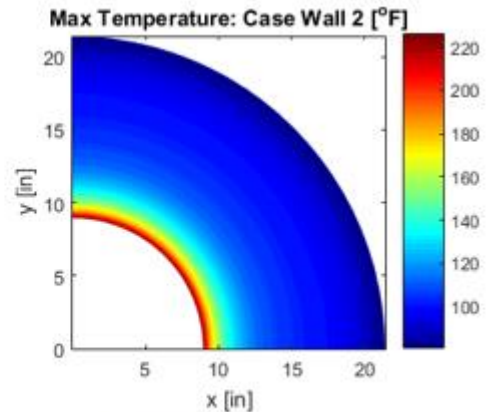
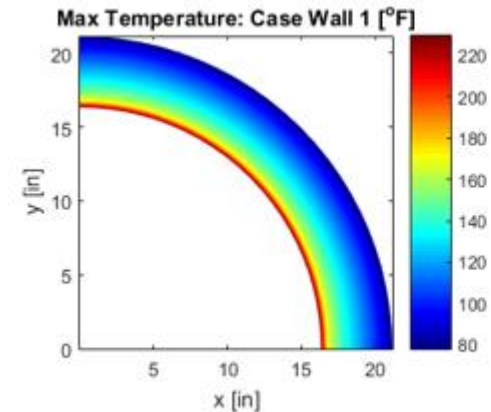
- Max T: ~ 1580 °F (~ 1100 °F neglecting inter-turbine duct)
- Max Increasing dT/dt : 5 °F/sec (neglecting nozzle)
- Max Decreasing dT/dt : 2.5 °F/sec (neglecting nozzle)
- Observations:
 - Max temperature occurs in the inter-turbine duct region due high temperatures from the aggressive cycle design and lack of active cooling
 - Max temperatures for different locations are shown to occur during climb, cruise, and heat soak





Application: Results

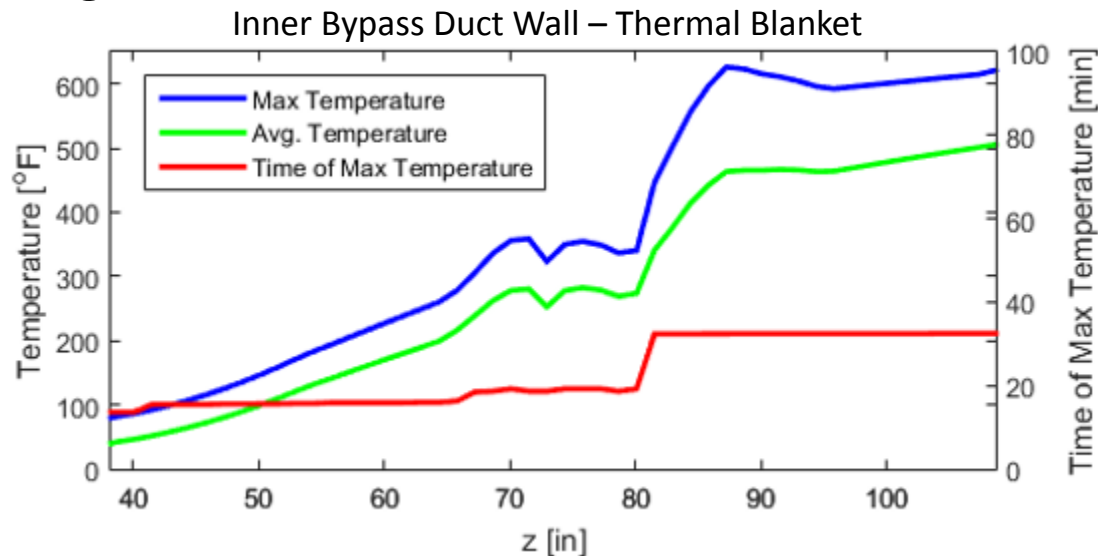
- Max T: ~220 °F
- Max Increasing dT/dt: ~1.3 °F/sec (at engine case), ~0.3 °F/sec (away from the engine case)
- Max Decreasing dT/dt: ~0.7 °F/sec (at engine case), ~0.1 °F/sec (away from the engine case)
- Observations:
 - Max temperature occur during heat soak, and to a lesser extent take-off





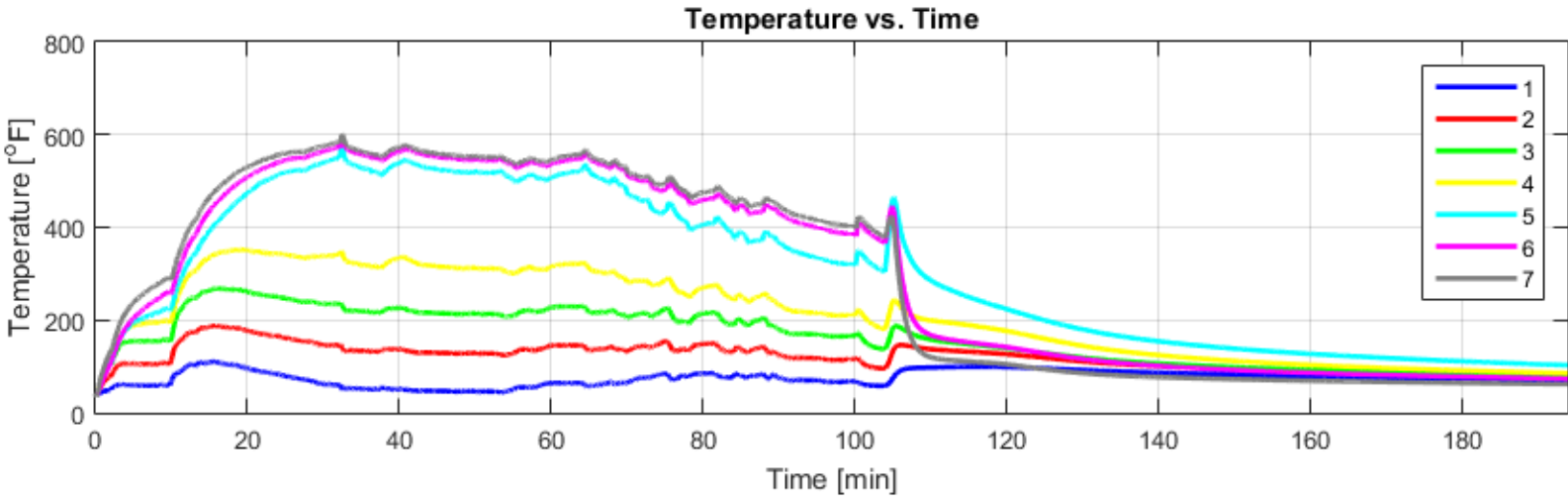
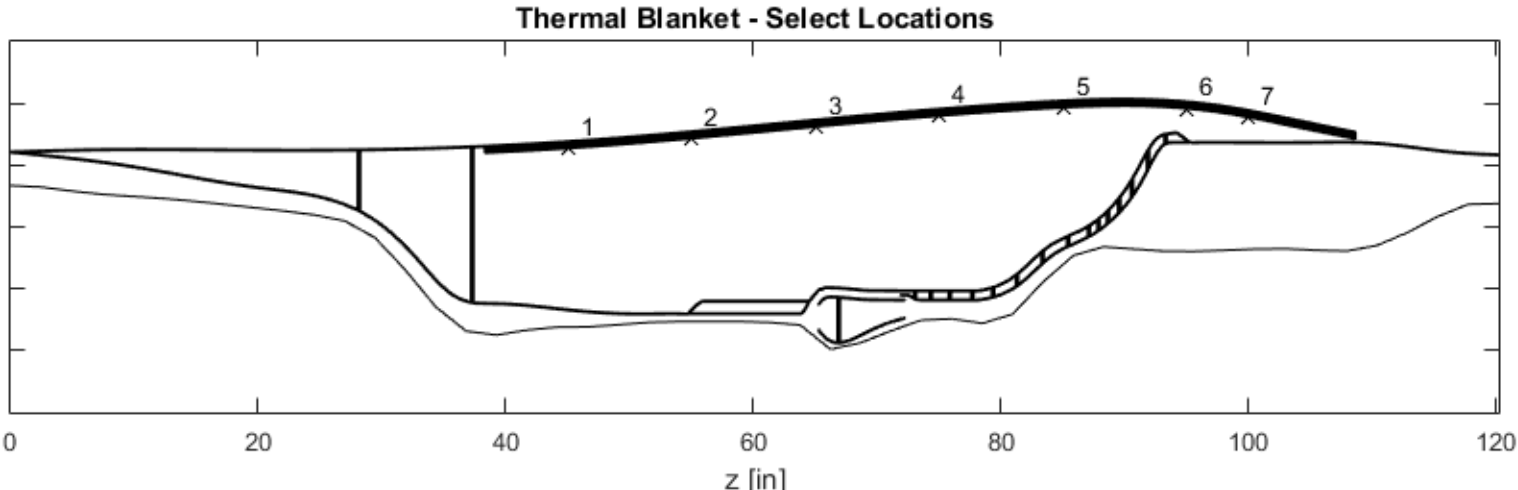
Application: Results

- Max T: ~ 600 °F (~ 350 °F for the region upstream of the inter-turbine duct)
- Max Increasing dT/dt : ~ 2 °F/sec
- Max Decreasing dT/dt : ~ 1.5 °F/sec
- Observations:
 - Max temperatures reached during climb to shortly after reaching cruise





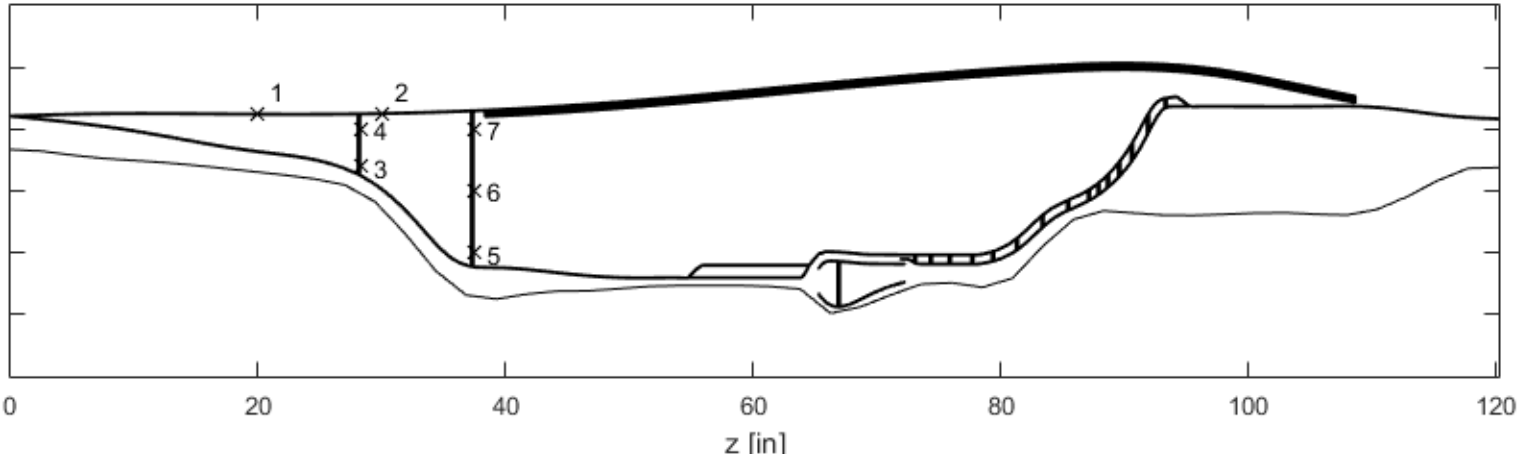
Application: Results



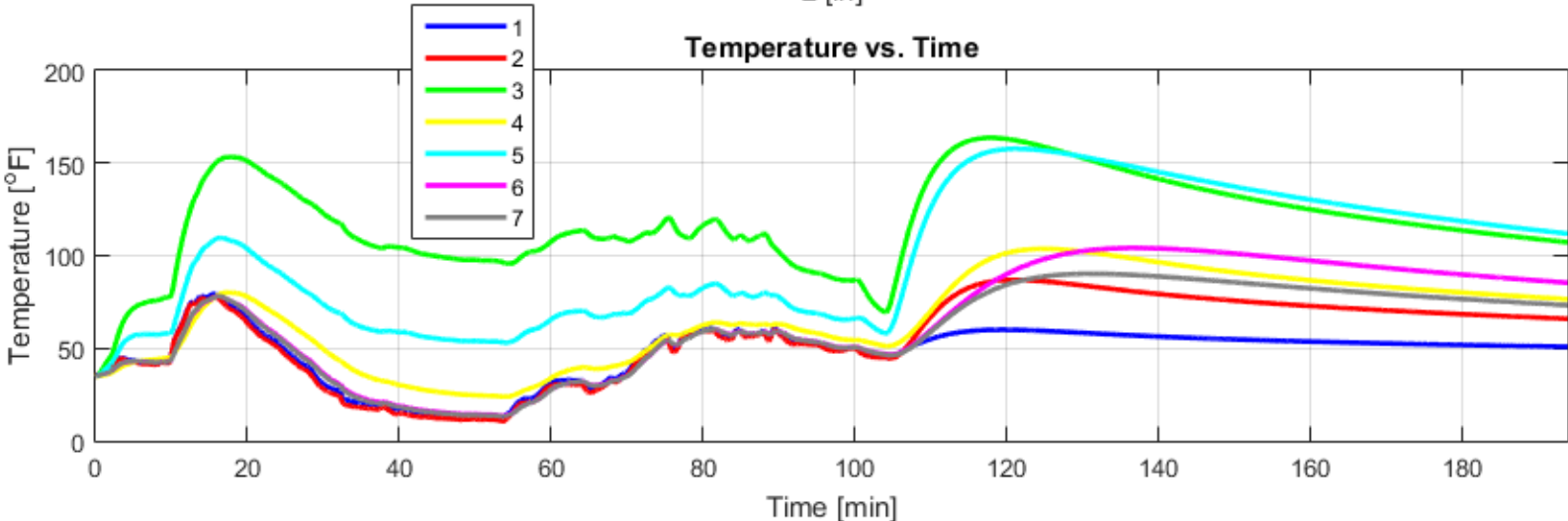


Application: Results

Front Inner Duct Wall & Case Walls: Select Locations

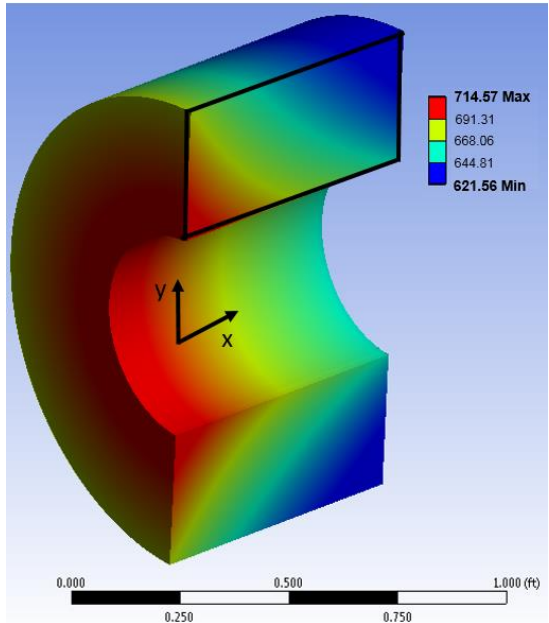
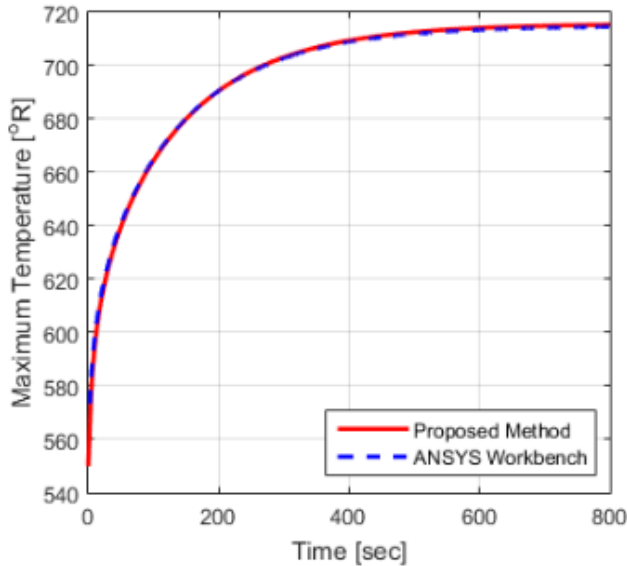
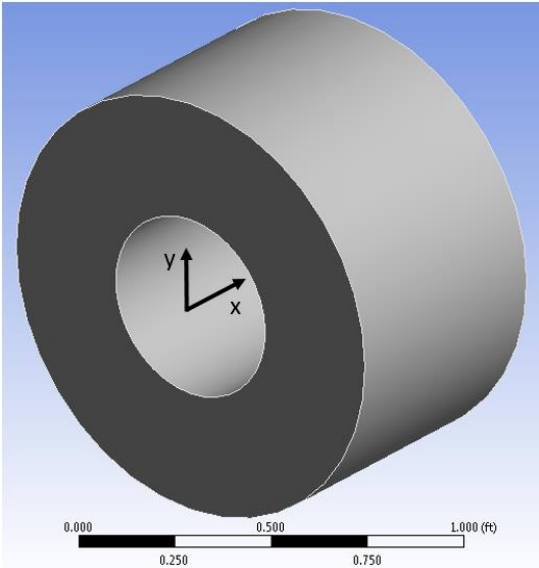


Temperature vs. Time

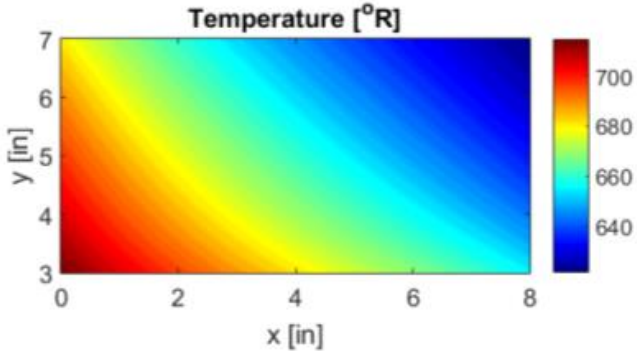




Extra Slides: ANSYS Verification Example



Proposed Method →



↑
ANSYS Workbench