



# PAT Wing Fixed Base Correction Modal Testing Debrief



Kevin Napolitano  
ATA Engineering

Natalie Spivey  
Kia Miller  
NASA Armstrong Flight Research Center

**Prepared for:**

NESC Meeting

**Date:**


**August 23, 2018**

13290 Evening Creek Drive S, Suite 250, San Diego CA 92128

 (858) 480-2000

 [www.ata-e.com](http://www.ata-e.com)

 [ata-engineering](https://www.linkedin.com/company/ata-engineering)

 [@ATAEngineering](https://twitter.com/ATAEngineering)

# Presentation Outline

---

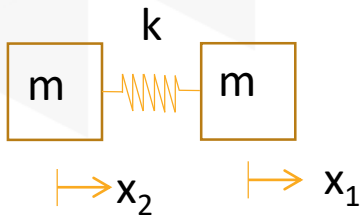
**Note: All results shown here are preliminary**

The objective of this presentation is to describe the PAT Wing modal testing conducted and results using Fixed Base Correction (FBC) method.

- FBC Theory – Drive point accelerations as references
- Description of Modal Test Setup
- FBC Signal Processing Techniques
  - FRF calculation
  - Partial FRF matrix inversion technique (SMURF)
- Decisions on where to add shakers (if needed)
- FBC Results

# Fixed Base Correction - Theory

- Original fixed base correction theory used constraint shapes as references, but the method is more easily deployed **using drive point accelerations as references**



$$\begin{bmatrix} s^2m + k & -k \\ -k & s^2m + k \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} F_1 \\ F_2 \end{Bmatrix}$$

$F_1$  and  $F_2$  as References:  $\omega_n^2 = 0, 2k/m$

$$X_1 = \frac{s^2m_2 + k}{(s^2m + k)(s^2m + k) - k^2} F_1 + \frac{k}{(s^2m + k)(s^2m + k) - k^2} F_2$$

$$X_2 = \frac{k}{(s^2m + k)(s^2m + k) - k^2} F_1 + \frac{s^2m_2 + k}{(s^2m + k)(s^2m + k) - k^2} F_2$$

$F_1$  and  $X_2$  as References:  $\omega_n^2 = k/m$

$$X_1 = \frac{1}{(s^2m + k)} F_1 + \frac{k}{(s^2m + k)} X_2$$

$$X_2 = 0F_1 + 1X_2$$

- **Using a drive point accelerometer as reference results in system modes with that DOF fixed**
- Make sure drive point FRF are as co-located as practicable
- Make sure drive point FRF are as clean as practicable
  - Use seismic accelerometers as drive points
  - Drive base shakers harder than wingtip shakers

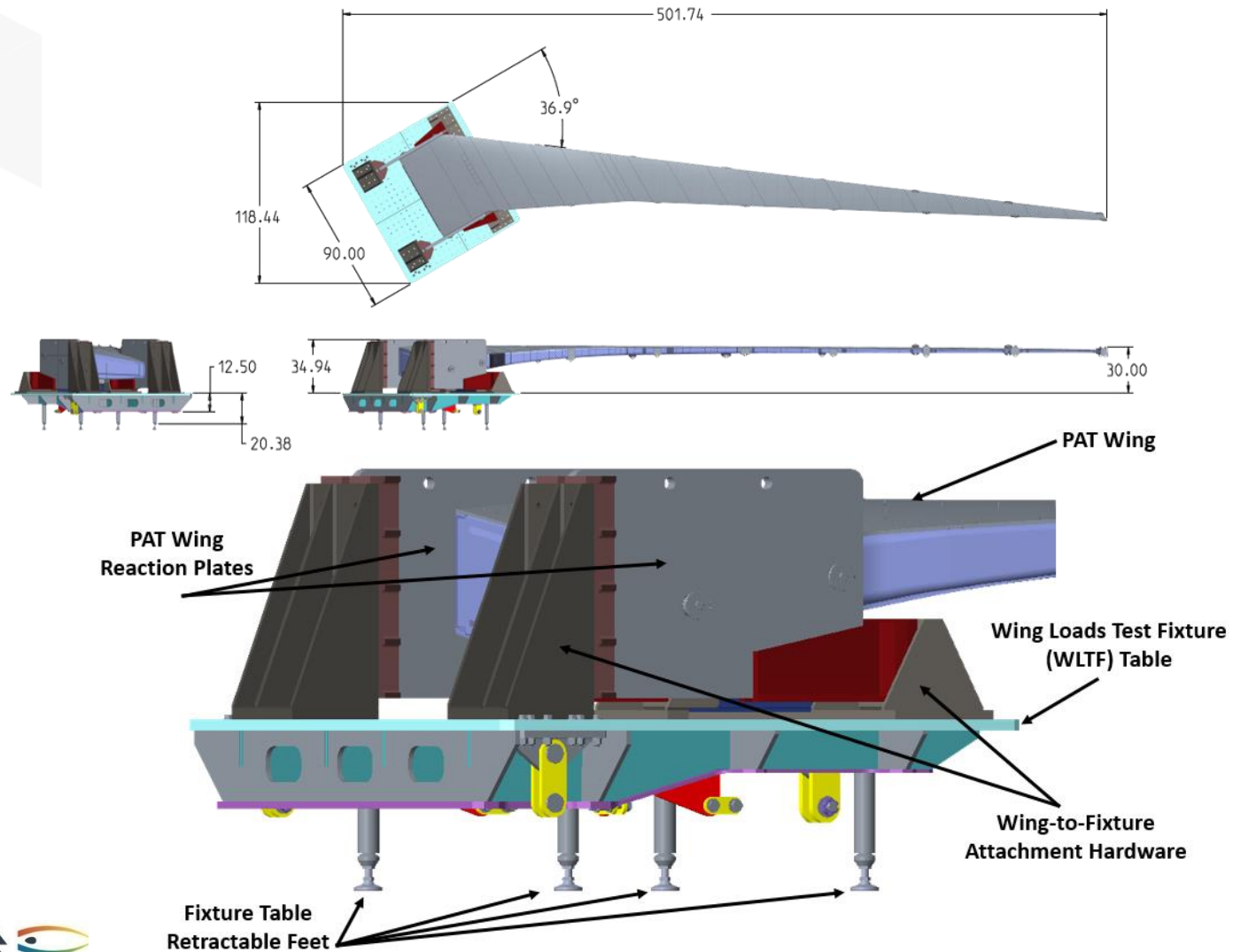
# PAT Wing Test Article

- Passive Aeroelastic Tailored (PAT) Wing is a tow-steered graphite epoxy, high aspect ratio, semi-span ( $\approx 39\text{ft}$ ) right wing box
- Designed and built for NASA by Aurora Flight Sciences
- Project funded by NASA ARMD Advanced Air Transport Technology (AATT) Project



# PAT Wing Test Setup

**Test Goal: Extract all fixed base modes to first torsion mode (near 55 Hz)**



# Modal Test Physical Boundary Conditions

**Using these boundary conditions would lead to a very challenging model updating effort without Fixed Base Correction**

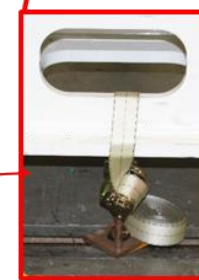
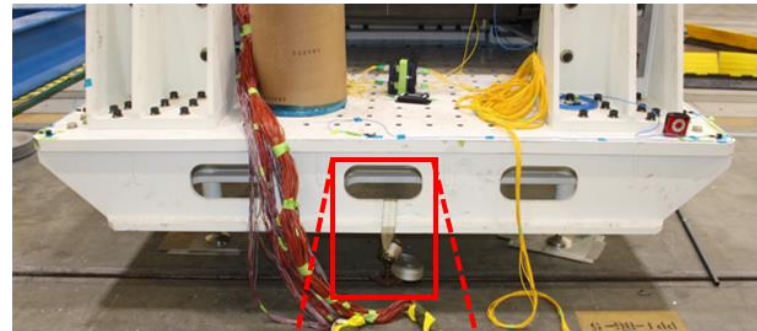
- Test physical boundary conditions: White static test fixture on the lab floor with four retractable feet and one location on the fixture that is secured to the lab floor with a strap
  - Dynamically active boundary condition

## Static Test Fixture Boundary Condition on Lab Floor



Four Retractable Feet  
Boundary Condition

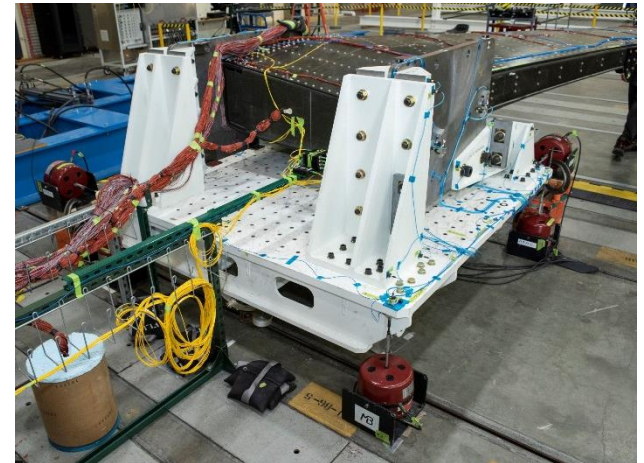
Table Secured with Strap to FLL Floor  
Boundary Condition



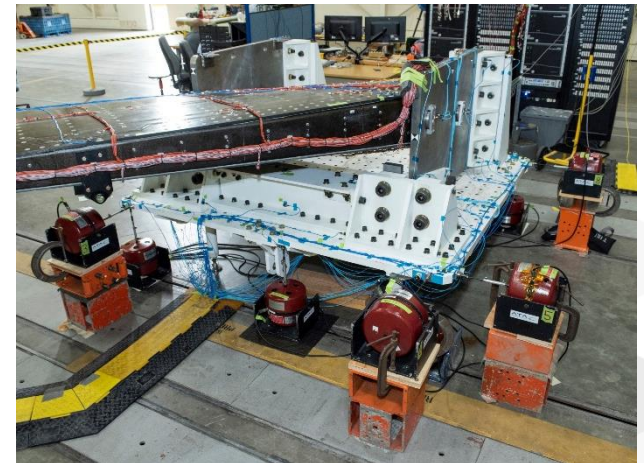
# Initial Modal Test Setup

- ≈ 270 Accelerometers
- 10 Shakers/load cells
  - 10 Seismic accels (drive points)
- True Random Input
  - Flexibility in signal processing

**Config. 1: 9 Base Shakers**



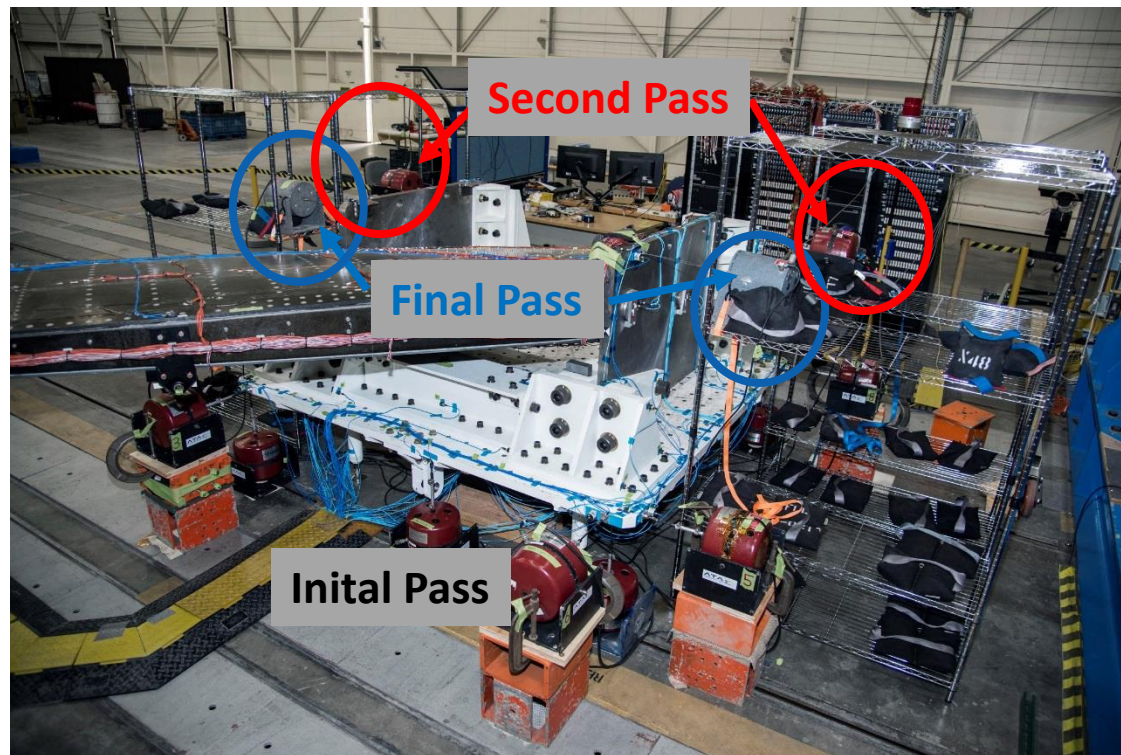
**Wingtip Shaker**



# Three Shaker Configurations

- Config. 1 (Initial Pass): 10 shakers – 9 on white static test fixture, 1 on wingtip
- Config. 2 (Second Pass): 12 shakers – Added 2 to aft white triangular brackets (lateral)
- Config. 3 (Final Pass): 14 shakers – Added 2 more to fwd wing root metallic plates (lateral)

## Shaker Layout





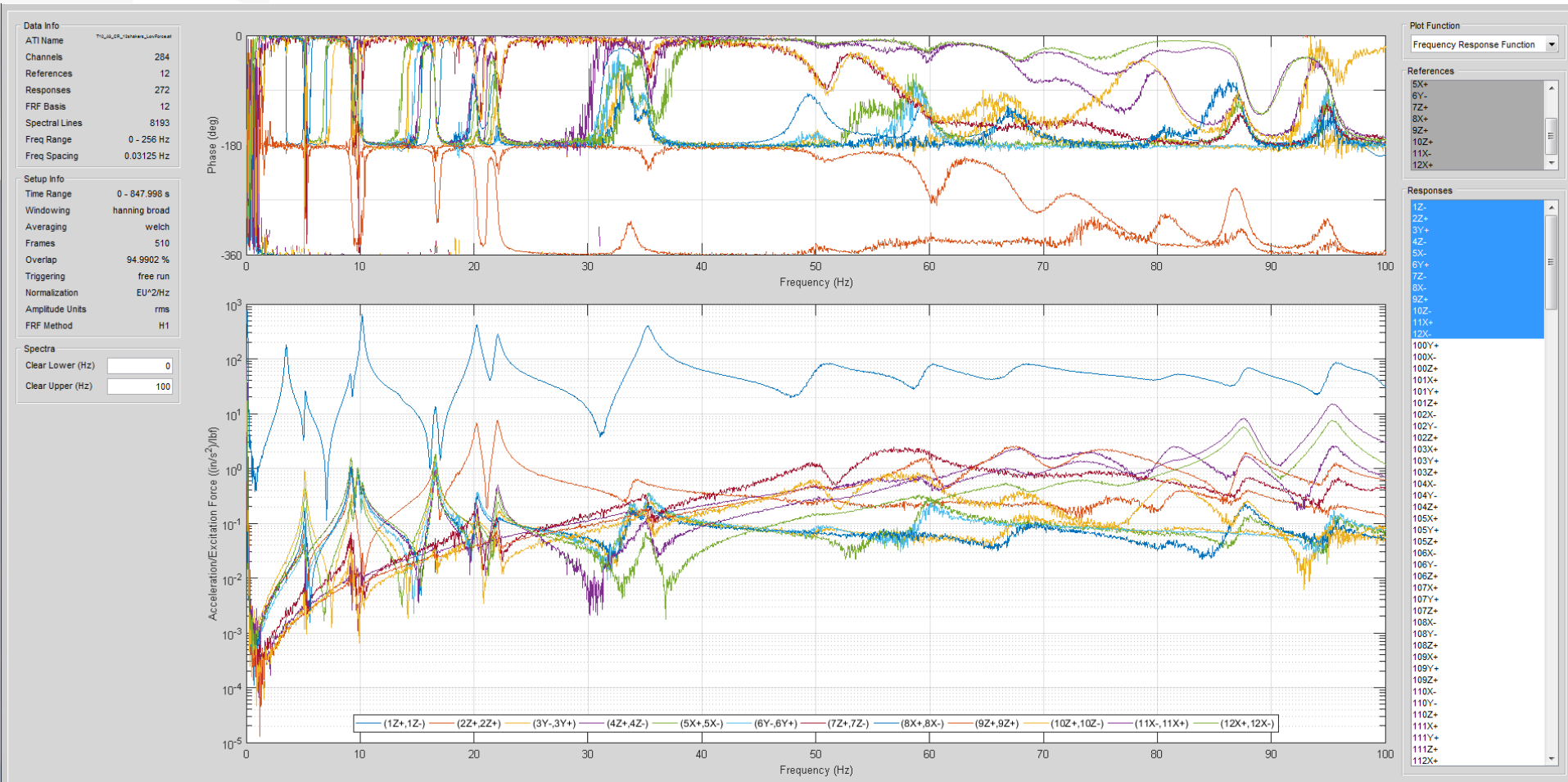
# Fixed Base Correction - Signal Processing is Iterative

- Forces usually have the highest S/N ratio and are therefore usually the best choice as basis vectors
  - Step 1. Calculate A/F FRF
  - Step 2. Use IMAT **SMURF** function to perform partial inversion of FRF matrix
    - Example:  $g = \text{smurf}(f, \text{ref}(2:\text{end}), \text{ref}(2:\text{end}))$ , where
      - F is the FRFs in all positive DOF ( $f = \text{fn2pos}(f)$ )
        - Not strictly necessary, but easier to manage
      - REF is the reference dof ( $[\text{ref}, \text{res}] = \text{ref\_res}(f)$ )
    - Step 3. Review FRF, and try new signal processing parameters if needed
    - Step 4. Fit modes using FRF associated with wingtip shaker FRF

# Config 2: 12 Shakers

Filename: T10\_A3\_CR\_12shakers\_LowForce.atf

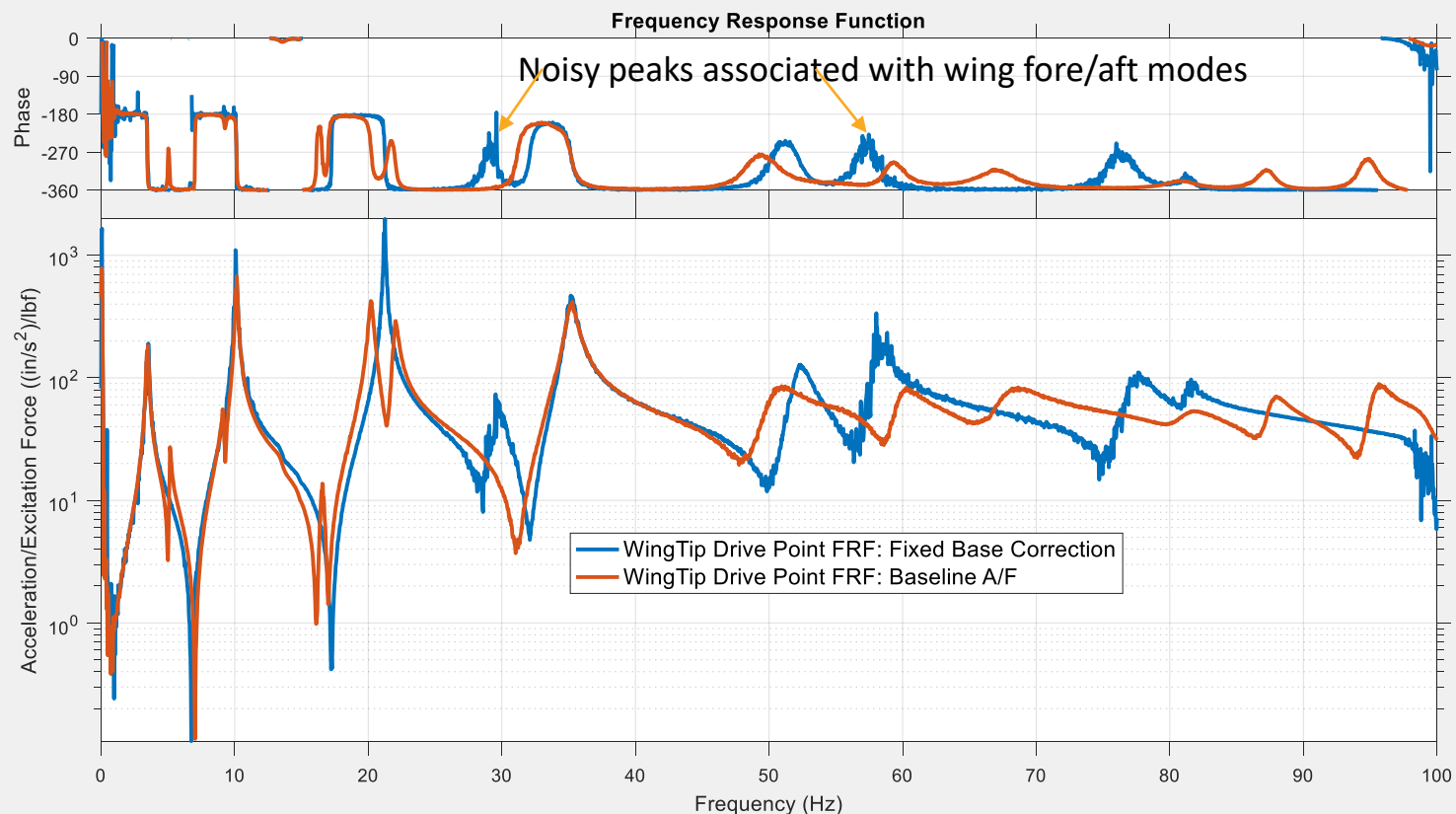
➤ Step 1: Calculate A/F FRF (using ALL shaker load cells)



# Config 2: Use SMURF on Fixture Drive Points

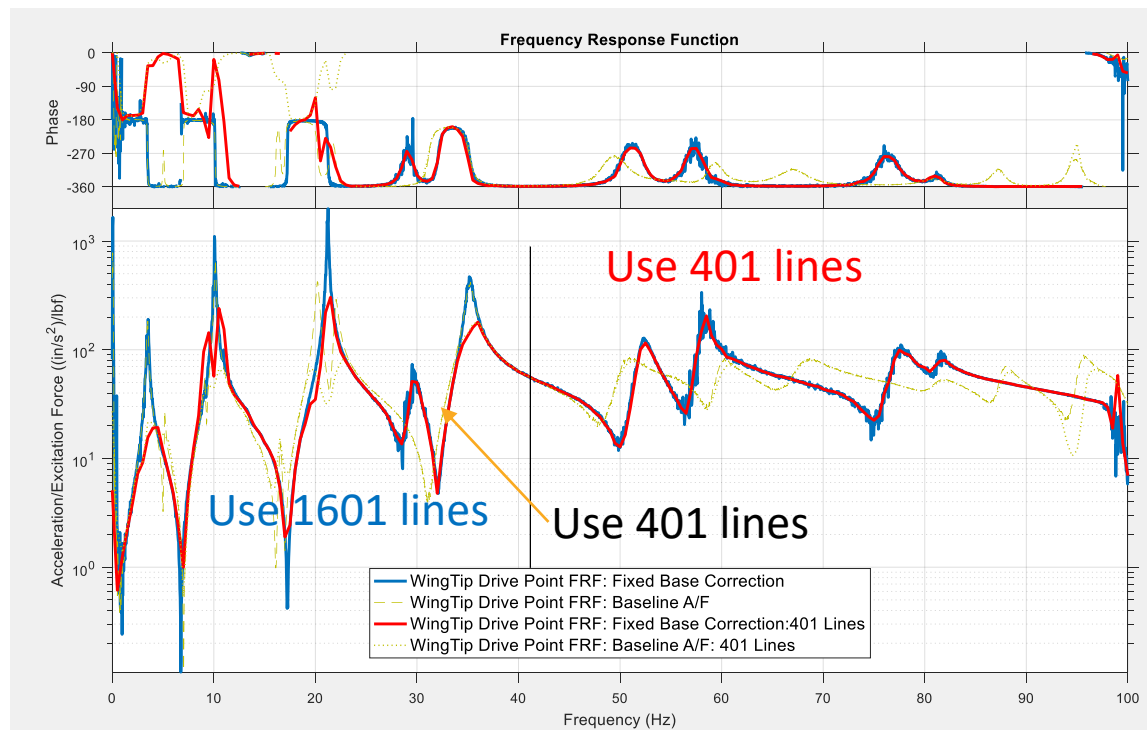
```
g=smurf(f,ref(2:end),ref(2:end))
```

- Step 2: Partially invert FRF matrix so base drive point accelerometers and wingtip shaker forces are references using SMURF technique. (Use MATLAB IMAT function)



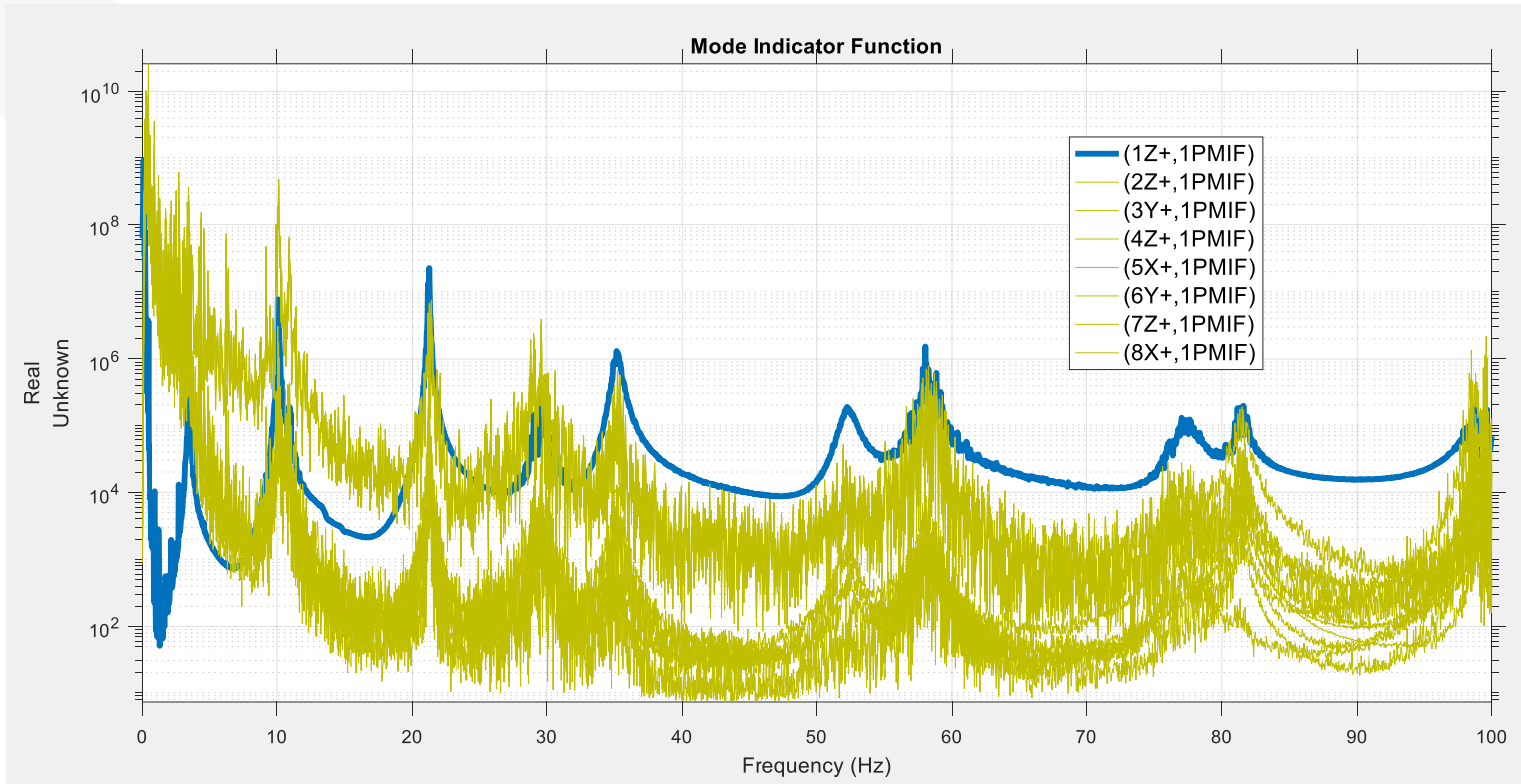
# Config 2: Increasing $\Delta f$ can help clean up FRF

- Step 3: Try different signal processing parameters when needed
- Optimal signal processing parameters may vary with frequency
- Takeaway: Signal processing is iterative



## Config 2: A/A FRF are Usually Noisy

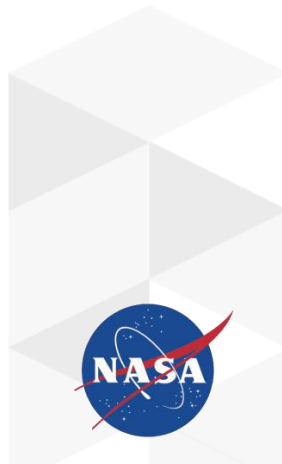
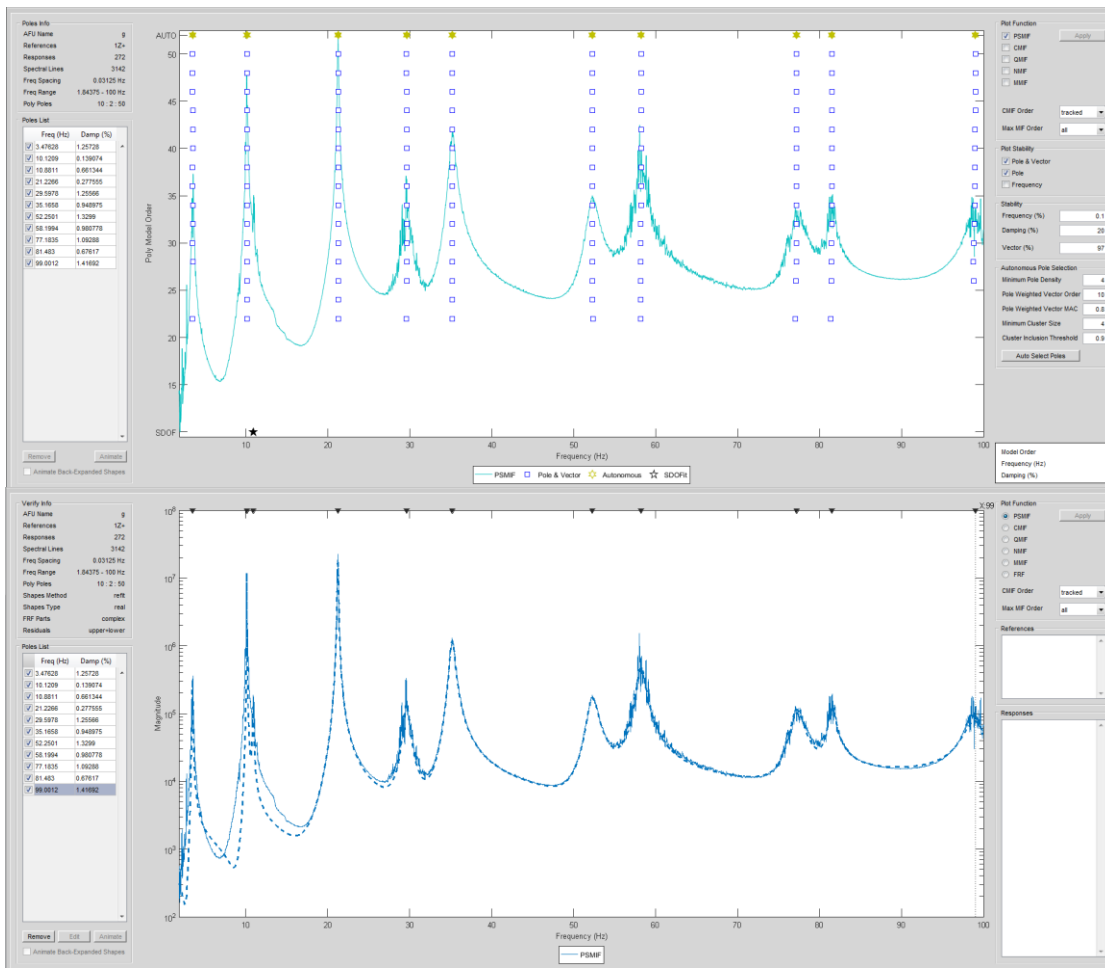
Use A/F FRF to fit modes



- Base structure A/A can be cleaned up by using constraint shapes as references, but doing so takes effort and most often the structure A/F FRF are sufficient to extract modes

# Config 2: Use FRFs associated with Wingtip Shaker Force to Fit Modes

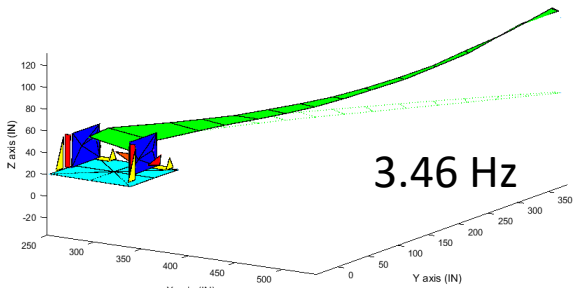
➤ Step 4: Fit modes using standard modal analysis software



# Config 2: 12 Shakers Results

Mode 1  
Frequency: 3.464 Hz  
Damping: 1.145 %Cr  
IDLine 1: Matlab Workspace Variable g  
IDLine 4: T10\_A3\_CR\_12shakers\_lowForce

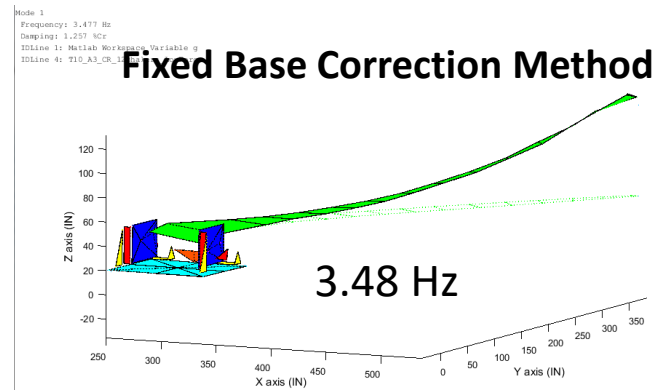
## Baseline / Nominal Method



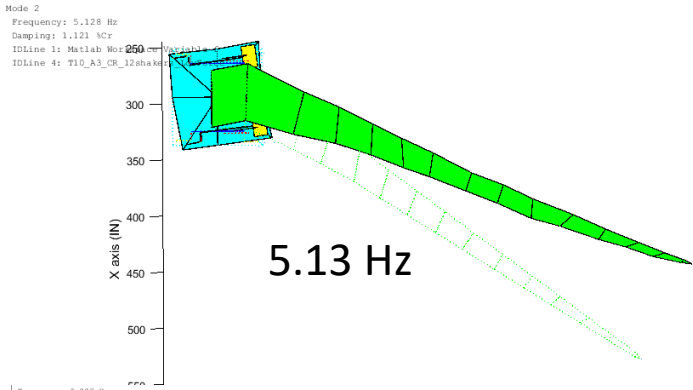
3.46 Hz



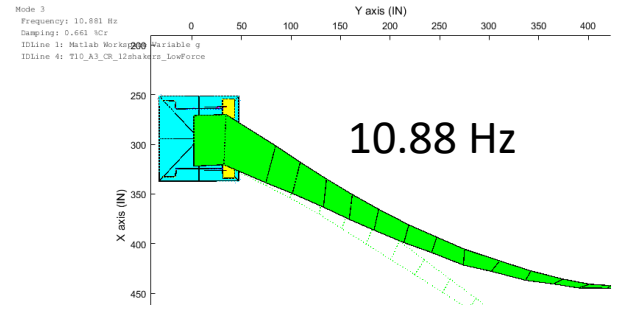
## Fixed Base Correction Method



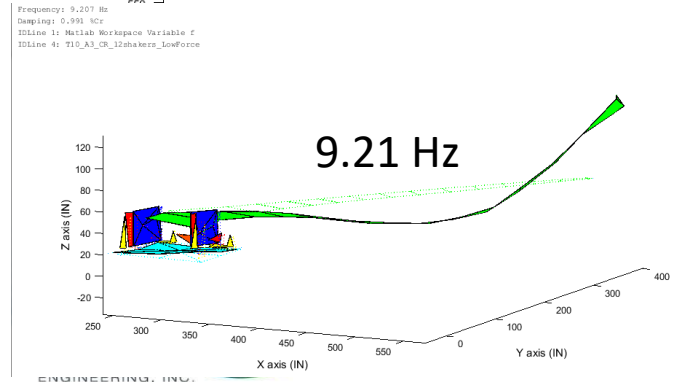
3.48 Hz



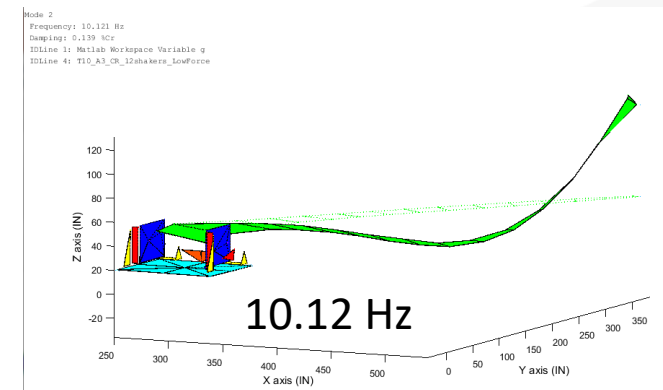
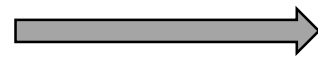
5.13 Hz



10.88 Hz



9.21 Hz

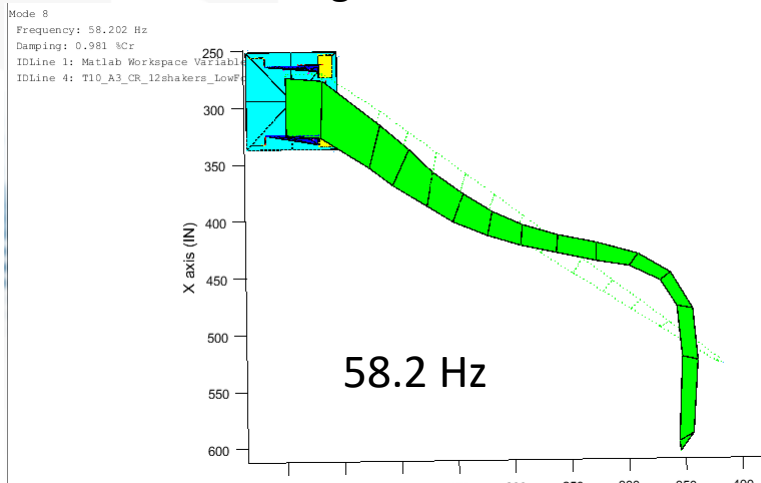


10.12 Hz

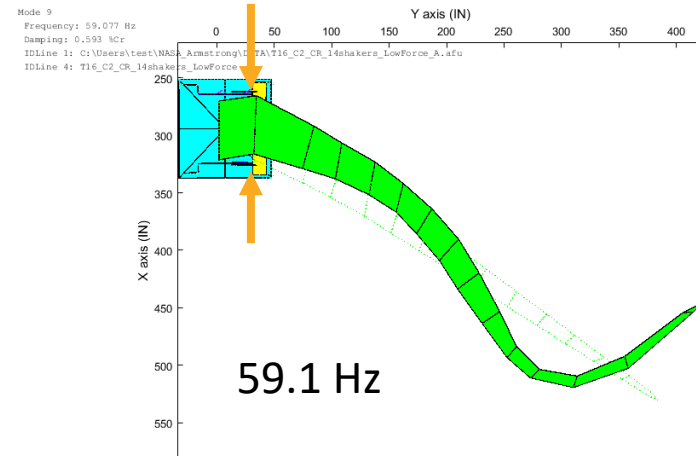
# Config 3: 14 Shakers Results

Helps remove flexible motion in wing root metallic plates

## Fixed Base Correction Method Config 2: 12 Shakers



## Fixed Base Correction Method Config 3: 14 Shakers

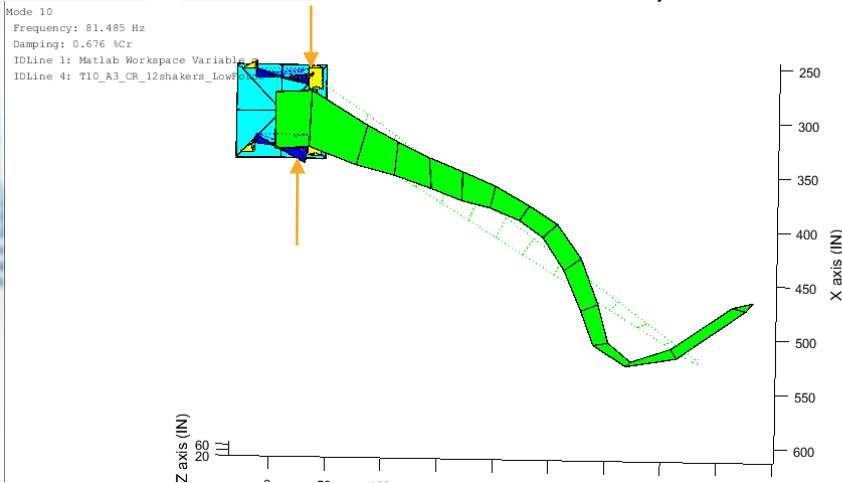




# Config 3: 14 Shakers Results

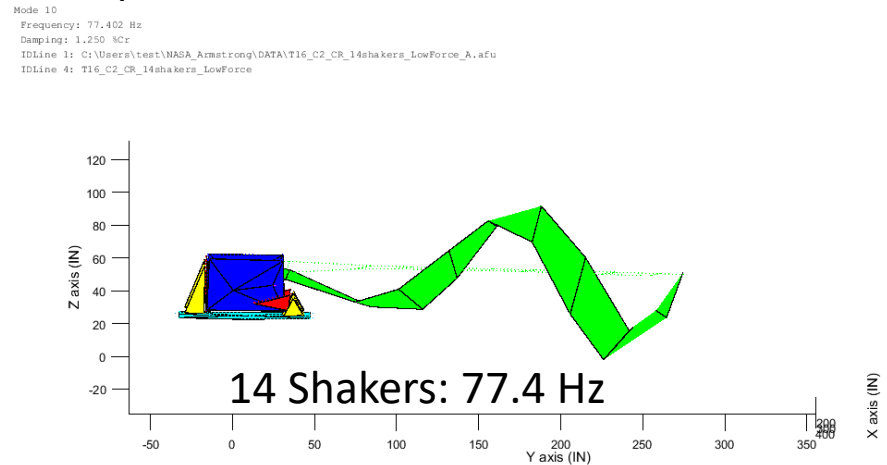
Two additional shakers removes lateral motion, but opens up new concerns

“Infinite Loop” of removing compliance by adding more shakers is limited by time, number of available shakers, or test objective requirements.



12 Shakers: 81.5 Hz

81 Hz mode with 12 shakers is eliminated with 14 shakers



Some vertical pedestal motion at 74 Hz with 14 shakers  
Could mount vertical shakers to remove this mode,  
however:

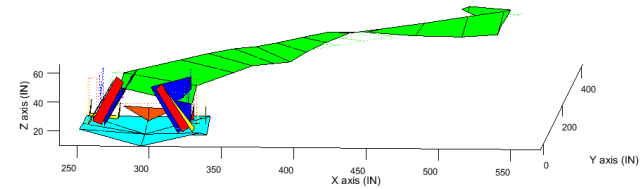
1. This mode is higher in frequency than last target mode
2. We ran out of source signal generators

# Config 1: 10 Shakers

FEM assumes white static test fixture is fixed

Test: 88.99 Hz  
White Triangular Stiffbacks Bending

Mode 10  
Frequency: 88.987 Hz  
Damping: 0.002 NCS



Test Self MAC Table

Test Shapes

Test Shapes	1	2	3	4	5	6	7	8	9	10	11
1	3.48	10.09	10.77	21.21	29.40	35.14	52.22	57.70	75.66	88.99	98.57
2	10.09	100		38	12						
3	10.77		100		29			8		5	6
4	21.21	14	38	100		36	7				
5	29.40		29		100			38	9	10	5
6	35.14	6	12		36	100	24				
7	52.22			7		24	100		6		
8	57.70		8		38			100	27	21	
9	75.66				9		6	27	100	23	
10	88.99		5		10			21	23	100	
11	98.57		6		5						100

Test Mode No.	FEM Mode No.	Test Freq (Hz)	FEM Freq (Hz)	Freq Pct Diff	Cross MAC	CRS MAC 3%	CRS MAC All
1	1	3.48	3.38	-2.8	98	99	99
2	2	10.09	10.33	2.3	98	99	100
3	3	10.77	10.95	1.7	92	96	97
4	4	21.21	22.36	5.4	98	99	100
5	5	29.40	29.32	-0.3	98	99	99
6	6	35.14	36.61	4.2	96	98	99
7	8	52.22	51.67	-1.1	65	81	99
8	11	57.70	62.18	7.8	82	91	99
9	12	75.66	73.43	-3.0	65	80	99
10	16	88.99	99.65	12.0	49	70	95
11	14	98.57	92.77	-5.9	91	95	99

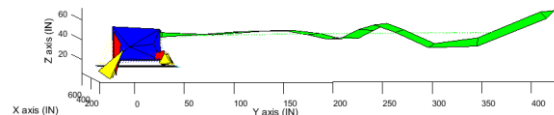
FEM/Test Cross MAC Table

FEM Shapes

Test Shapes	FEM Shapes																	Test	Test	
	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	CRSS 3%	CRSS All	
1	3.38	98	31																99	99
2	10.09	32	98	41															99	100
3	10.77		92	24															96	97
4	21.21		33	98	44														99	100
5	29.40			39	98	46				29									99	99
6	35.14				31		96												98	99
7	52.22					21		65	25			24							81	99
8	57.70				38		65			38	82	20				23			91	99
9	75.66									27	50	65				23	32		80	99
10	88.99										39	27					49		70	95
11	98.57																	91	95	99
CRSS 3%	99	99	96	99	99	98	81	81	50	61	90	80	49	95	48	70	32			
CRSS All	99	100	97	100	99	99	95	84	64	80	97	83	80	98	69	76	52			

Mode 10  
Frequency: 61.192 Hz  
Damping: 0.000 NCS

FEM: 61.19 Hz  
Triangular Bracket



# Config 2: 12 Shakers

FEM assumes white static test fixture & four aft triangular stiffbacks are fixed

Test Self MAC Table

Test Shapes

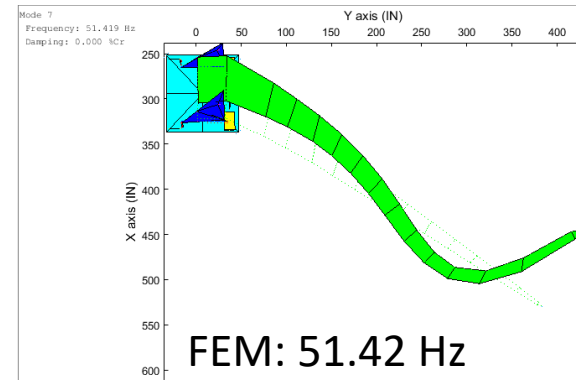
Test Shapes	1	2	3	4	5	6	7	8	9	10	11
1	3.48	100	32	14	6						
2	10.12	32	100	38	12						
3	10.90			100	34	10	5	9			
4	21.23	14	38	100	36	7					
5	29.59			34	100	38	16	9			
6	35.17	6	12	36	100	24					
7	52.25			7	24	100	20	5			
8	58.20			10	38	100	36				
9	77.22					20	100				
10	81.50			5	16	5	36	100			
11	99.42			9	9						100

Test Mode No.	FEM Mode No.	Test Freq (Hz)	FEM Freq (Hz)	Freq Pct Diff	Cross MAC	CRS XMAC 3%	CRS XMAC All
1	1	3.48	3.38	-2.9	99	99	100
2	2	10.12	10.36	2.4	99	99	100
3	3	10.90	11.16	2.4	97	99	100
4	4	21.23	22.40	5.5	98	99	100
5	5	29.59	30.83	4.2	97	98	99
6	6	35.17	36.90	4.9	96	98	99
7	8	52.25	51.78	-0.9	54	80	99
8	10	58.20	67.82	16.5	73	85	99
9	11	77.22	76.23	-1.3	63	79	98
10	10	81.50	67.82	-16.8	64	80	99
11	12	99.42	93.15	-6.3	95	97	99

FEM/Test Cross MAC Table

FEM Shapes

Test Shapes	1	2	3	4	5	6	7	8	9	10	11	12	13	Test CRSS	Test All
1	3.38	10.36	11.16	22.40	30.83	36.90	51.42	51.78	55.10	67.82	76.23	93.15	100.98	3%	All
2	3.48	99	31											99	100
3	10.12	32	99	41										99	100
4	10.90			97	27									99	100
5	21.23		34		98	43								99	100
6	29.59			41		97	31			25				98	99
7	35.17				31		96			23				98	99
8	52.25							26	54	28	26			80	99
9	58.20				43			62			73			85	99
10	77.22											63	31	79	98
11	81.50													80	99
12	99.42													97	99
CRSS 3%	99	99	99	99	98	98	78	74	53	85	79	97	55		
CRSS All	100	100	100	100	99	99	95	82	65	99	99	99	64		



FEM: 51.42 Hz  
Lateral wing root metallic plate mode

# Config 3: 14 Shakers

FEM assumes everything but wing is fixed

**Test Self MAC Table**  
**Test Shapes**

Test Shapes	1	2	3	4	5	6	7	8	9	10	11	12
1	3.48	10.05	11.02	21.22	30.15	35.23	52.20	56.67	59.08	77.40	98.45	#####
2	3.48	100	30		13		6					
3	10.05	30	100		37		12					
4	11.02			100		33			8		8	
5	21.22	13	37		100		35	9				
6	30.15				33	100			37		8	
7	35.23	6	12			35	100	25		5		
8	52.20			9		25	100			22		8
9	56.67							100			8	
10	59.08			8		37			100			
11	77.40						5	22		100		21
12	98.45			8	8						100	
12	106.17						8			21		100

Test Mode No.	FEM Mode No.	Test Freq (Hz)	FEM Freq (Hz)	Freq Pct Diff	Cross Ortho	CRS XOrtho 3%	CRS XOrtho All
1	1	3.48	3.39	-2.4	99	99	100
2	2	10.05	10.45	4.0	98	99	100
3	3	11.02	11.35	3.0	97	98	100
4	4	21.22	22.64	6.7	99	99	100
5	5	30.15	31.98	6.1	96	98	100
6	6	35.23	37.46	6.3	96	98	100
7	7	52.20	51.94	-0.5	62	79	100
8	8	56.67	55.54	-2.0	57	75	100
9	9	59.08	65.92	11.6	79	89	99
10	10	77.40	77.43	0.0	83	91	99
11	11	98.45	93.69	-4.8	95	97	100
12	12	106.17	102.17	-3.8	74	86	99

**FEM/Test Cross MAC Table**  
**FEM Shapes**

Test Shapes	1	2	3	4	5	6	7	8	9	10	11	12	Test CRSS 3%	Test CRSS All
1	3.39	10.45	11.35	22.64	31.98	37.46	51.94	55.54	65.92	77.43	93.69	102.17	99	100
2	3.48	99	29										99	100
3	10.05	31	98	40									99	100
4	11.02			97	26								98	100
5	21.22		35		99	41							99	100
6	30.15			41		96			25				98	100
7	35.23				31		96	20					98	100
8	52.20						20	62	29	22			79	100
9	56.67							42	57				75	100
10	59.08				46					79			89	99
11	77.40										83	22	91	99
12	98.45											95	97	100
12	106.17												74	86
CRSS 3%	99	99	98	99	98	98	79	75	89	91	97	86		
CRSS All	100	100	100	100	99	99	100	99	96	96	99	92		

Mode 9  
Frequency: 59.07 Hz  
Damping: 0.593 NCR  
IDLine 1: C:\Users\test\NASA\Armet\com\DATA\T16\_C2\_CR\_14shakers\_lowForce\_A.afu  
IDLine 4: T16\_C2\_CR\_14shakers\_lowForce\_A.afu



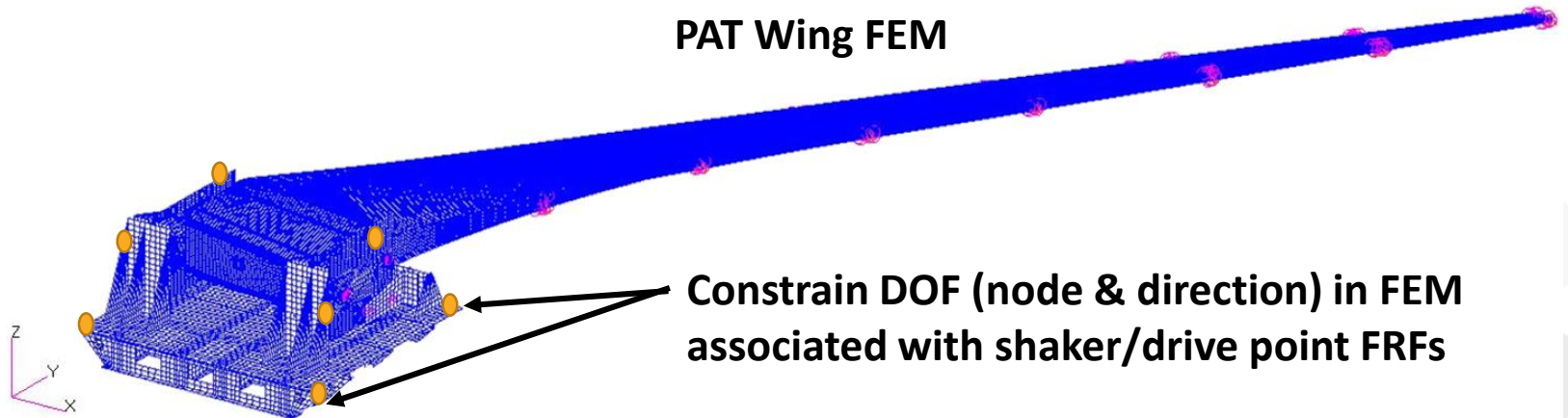
Test: 59.08 Hz  
Fore-Aft mode, Pin joint slip

# Examples of Mode Shapes

- View Animations of Fixed Base Modes
- Baseline A/F FRF had significant base motion
- Initial shaker set resulted in bending mode of base at 80+ Hz
- Second shaker set removed base bending motion
- Sliding motion of wing root pinned connection amplified when 14 shaker set used

# Model Updating Strategy

- Use full finite element model (FEM), but constrain DOF associated with shaker/drive point accelerometers to best match the testing results
  - May have to spread load to avoid stress concentrations & local deformation of “point load” due to single DOF constraint



# Test Summary

- Fixed Base Correction method was successfully used to extract fixed base modal results for the PAT wing that was mounted to a dynamically active static test fixture resting unsecured on a test facility floor.
- There are many potential scenarios where this FBC method can be used on future tests of structures mounted on other dynamically active static test fixtures.