



This is a repository copy of *Relationships between the nonlinear output frequency response functions of multi- degree-of-freedom nonlinear systems.*

White Rose Research Online URL for this paper:
<http://eprints.whiterose.ac.uk/74584/>

Monograph:

Peng, Z.K., Lang, Z.Q. and Billings, S.A. (2006) Relationships between the nonlinear output frequency response functions of multi- degree-of-freedom nonlinear systems. Research Report. ACSE Research Report no. 929 . Automatic Control and Systems Engineering, University of Sheffield

Reuse

Unless indicated otherwise, fulltext items are protected by copyright with all rights reserved. The copyright exception in section 29 of the Copyright, Designs and Patents Act 1988 allows the making of a single copy solely for the purpose of non-commercial research or private study within the limits of fair dealing. The publisher or other rights-holder may allow further reproduction and re-use of this version - refer to the White Rose Research Online record for this item. Where records identify the publisher as the copyright holder, users can verify any specific terms of use on the publisher's website.

Takedown

If you consider content in White Rose Research Online to be in breach of UK law, please notify us by emailing eprints@whiterose.ac.uk including the URL of the record and the reason for the withdrawal request.



eprints@whiterose.ac.uk
<https://eprints.whiterose.ac.uk/>

Relationships between the Nonlinear Output Frequency Response Functions of Multi-Degree-of-Freedom Nonlinear Systems

Z K Peng, Z Q Lang, and S A Billings



Department of Automatic Control and Systems Engineering
The University of Sheffield
Mappin Street, Sheffield
S1 3JD, UK

Research Report No. 929

July 2006

Relationships between the Nonlinear Output Frequency Response Functions of Multi-Degree-of-Freedom Nonlinear Systems

Z.K. Peng, Z.Q. Lang, and S. A. Billings

Department of Automatic Control and Systems Engineering, University of Sheffield
Mappin Street, Sheffield, S1 3JD, UK

Email: z.peng@sheffield.ac.uk; z.lang@sheffield.ac.uk; s.billings@sheffield.ac.uk

Abstract: Nonlinear Output Frequency Response Functions (NOFRFs) are a new concept proposed by the authors for the analysis of nonlinear systems in the frequency domain. The present study is concerned with investigating inherent relationships between the NOFRFs of two masses in nonlinear MDOF systems. The results reveal very important properties of a class of nonlinear systems, and have considerable significance for the application of the NOFRF concept in engineering practices.

1 Introduction

Linear systems, which have been widely studied by practitioners in many different fields, have provided a basis for the development of the majority of control system synthesis, mechanical system analysis and design, and signal processing methods. However, there are many qualitative behaviors in engineering, such as the generation of harmonics and inter-modulations, which cannot be produced by linear models [1]. In these cases, nonlinear models are needed to describe the system, and nonlinear system analysis methods have to be applied to investigate the system dynamics.

The Volterra series approach [2] is a powerful tool for the analysis of nonlinear systems, which extends the familiar concept of the convolution integral for linear systems to a series of multi-dimensional convolution integrals. The Fourier transforms of the Volterra kernels are called Generalised Frequency Response Functions (GFRFs) [3], and can be considered as extensions of the linear Frequency Response Function (FRF) to the nonlinear case. If a differential equation or discrete-time model is available for a nonlinear system, the GFRFs can be determined using the algorithm in [4]~[6]. However, the GFRFs are much more complicated than the FRF. GFRFs are multidimensional functions [7][8], and can be difficult to measure, display and interpret in practice.

Recently, the novel concept known as Nonlinear Output Frequency Response Functions (NOFRFs) was proposed by the authors [9]. The concept can be considered to be an alternative extension of the FRF to the nonlinear case. NOFRFs are one dimensional functions of frequency, which allow the analysis of nonlinear systems in the frequency domain to be implemented in a manner similar to the analysis of linear systems and which provide great insight into the mechanisms which dominate important nonlinear behaviours.

Based on the GFRFs for MIMO system achieved in [2][10], most recently, the authors also extended the concept of NOFRFs for the MIMO Volterra nonlinear systems [11]. Although great efforts have been made to analyze nonlinear systems in the frequency domain, most studies, including both numerical and experimental studies, have tended to focus on nonlinear systems with a single degree of freedom.

In engineering practice, many mechanical and structural systems require more than one coordinates to describe the system behaviours. This implies a MDOF model is often needed to describe such systems. In addition, these systems may also behave nonlinearly due to nonlinear characteristics of some components within the systems. For example, a beam with breathing cracks behaves nonlinearly only because of the cracked elements inside the beam [12]. These nonlinear MDOF systems can be regarded as locally nonlinear MDOF systems. The present study is concerned with derivation of the inherent relationships between the NOFRFs of any two masses in locally nonlinear MDOF systems. The results reveal the important properties of nonlinear MDOF systems and have considerable significance for the application of the NOFRF concept in engineering practices.

2. Nonlinear Output Frequency Response Function

2.1 Nonlinear Output Frequency Response Functions under General Input

The definition of NOFRFs is based on the Volterra series theory of nonlinear systems. The Volterra series extends the well-known convolution integral description for linear systems to a series of multi-dimensional convolution integrals, which can be used to represent a wide class of nonlinear systems [3].

Consider the class of nonlinear systems which are stable at zero equilibrium and which can be described in the neighbourhood of the equilibrium by the Volterra series

$$y(t) = \sum_{n=1}^N \int_{-\infty}^{\infty} \cdots \int_{-\infty}^{\infty} h_n(\tau_1, \dots, \tau_n) \prod_{i=1}^n u(t - \tau_i) d\tau_i \quad (1)$$

where $y(t)$ and $u(t)$ are the output and input of the system, $h_n(\tau_1, \dots, \tau_n)$ is the n th order Volterra kernel, and N denotes the maximum order of the system nonlinearity. Lang and Billings [3] derived an expression for the output frequency response of this class of nonlinear systems to a general input. The result is

$$\begin{cases} Y(j\omega) = \sum_{n=1}^N Y_n(j\omega) & \text{for } \forall \omega \\ Y_n(j\omega) = \frac{1/\sqrt{n}}{(2\pi)^{n-1}} \int_{\omega_1+\dots+\omega_n=\omega} H_n(j\omega_1, \dots, j\omega_n) \prod_{i=1}^n U(j\omega_i) d\sigma_{n\omega} \end{cases} \quad (2)$$

This expression reveals how nonlinear mechanisms operate on the input spectra to produce the system output frequency response. In (2), $Y(j\omega)$ is the spectrum of the system output, $Y_n(j\omega)$ represents the n th order output frequency response of the system,

$$H_n(j\omega_1, \dots, j\omega_n) = \int_{-\infty}^{\infty} \dots \int_{-\infty}^{\infty} h_n(\tau_1, \dots, \tau_n) e^{-j(\omega_1\tau_1 + \dots + \omega_n\tau_n)} d\tau_1 \dots d\tau_n \quad (3)$$

is the n th order Generalised Frequency Response Function (GFRF) [3], and

$$\int_{\omega_1+\dots+\omega_n=\omega} H_n(j\omega_1, \dots, j\omega_n) \prod_{i=1}^n U(j\omega_i) d\sigma_{n\omega}$$

denotes the integration of $H_n(j\omega_1, \dots, j\omega_n) \prod_{i=1}^n U(j\omega_i)$ over the n -dimensional hyper-plane $\omega_1 + \dots + \omega_n = \omega$. Equation (2) is a natural extension of the well-known linear relationship $Y(j\omega) = H(j\omega)U(j\omega)$, where $H(j\omega)$ is the frequency response function, to the nonlinear case.

For linear systems, the possible output frequencies are the same as the frequencies in the input. For nonlinear systems described by equation (1), however, the relationship between the input and output frequencies is more complicated. Given the frequency range of an input, the output frequencies of system (1) can be determined using the explicit expression derived by Lang and Billings in [3].

Based on the above results for the output frequency response of nonlinear systems, a new concept known as the Nonlinear Output Frequency Response Function (NOFRF) was recently introduced by Lang and Billings [9]. The NOFRF is defined as

$$G_n(j\omega) = \frac{\int_{\omega_1+\dots+\omega_n=\omega} H_n(j\omega_1, \dots, j\omega_n) \prod_{i=1}^n U(j\omega_i) d\sigma_{n\omega}}{\int_{\omega_1+\dots+\omega_n=\omega} \prod_{i=1}^n U(j\omega_i) d\sigma_{n\omega}} \quad (4)$$

under the condition that

$$U_n(j\omega) = \int_{\omega_1+\dots+\omega_n=\omega} \prod_{i=1}^n U(j\omega_i) d\sigma_{n\omega} \neq 0 \quad (5)$$

Notice that $G_n(j\omega)$ is valid over the frequency range of $U_n(j\omega)$, which can be determined using the algorithm in [3].

By introducing the NOFRFs $G_n(j\omega)$, $n=1, \dots, N$, equation (2) can be written as

$$Y(j\omega) = \sum_{n=1}^N Y_n(j\omega) = \sum_{n=1}^N G_n(j\omega) U_n(j\omega) \quad (6)$$

which is similar to the description of the output frequency response for linear systems. The NOFRFs reflect a combined contribution of the system and the input to the system output frequency response behaviour. It can be seen from equation (4) that $G_n(j\omega)$ depends not only on H_n ($n=1, \dots, N$) but also on the input $U(j\omega)$. For a nonlinear system, the dynamical properties are determined by the GFRFs H_n ($n=1, \dots, N$). However, from equation (3) it can be seen that the GFRF is multidimensional [7][8], which can make the GFRFs difficult to measure, display and interpret in practice. According to equation (4), the NOFRF $G_n(j\omega)$ is a weighted sum of $H_n(j\omega_1, \dots, j\omega_n)$ over $\omega_1 + \dots + \omega_n = \omega$ with the weights depending on the test input. Therefore $G_n(j\omega)$ can be used as an alternative representation of the dynamical properties described by H_n . The most important property of the NOFRF $G_n(j\omega)$ is that it is one dimensional, and thus allows the analysis of nonlinear systems to be implemented in a convenient manner similar to the analysis of linear systems. Moreover, there is an effective algorithm [9] available which allows the estimation of the NOFRFs to be implemented directly using system input output data.

2.2 Nonlinear Output Frequency Response Functions under Harmonic Inputs

When system (1) is subject to a harmonic input

$$u(t) = A \cos(\omega_F t + \beta) \quad (7)$$

Lang and Billings [3] showed that equation (1) can be expressed as

$$Y(j\omega) = \sum_{n=1}^N Y_n(j\omega) = \sum_{n=1}^N \left(\frac{1}{2^n} \sum_{\omega_{k_1} + \dots + \omega_{k_n} = \omega} H_n(j\omega_{k_1}, \dots, j\omega_{k_n}) A(j\omega_{k_1}) \dots A(j\omega_{k_n}) \right) \quad (8)$$

where

$$A(j\omega_{k_i}) = \begin{cases} |A| e^{j \text{sign}(k) \beta} & \text{if } \omega_{k_i} \in \{k\omega_F, k = \pm 1\}, i = 1, \dots, n \\ 0 & \text{otherwise} \end{cases} \quad (9)$$

Define the frequency components of the n th order output of the system as Ω_n , then according to equation (8), the frequency components in the system output can be expressed as

$$\Omega = \bigcup_{n=1}^N \Omega_n \quad (10)$$

where Ω_n is determined by the set of frequencies

$$\{\omega = \omega_{k_1} + \dots + \omega_{k_n} \mid \omega_{k_i} = \pm \omega_F, i = 1, \dots, n\} \quad (11)$$

From equation (11), it is known that if all $\omega_{k_1}, \dots, \omega_{k_n}$ are taken as $-\omega_F$, then $\omega = -n\omega_F$. If k of these are taken as ω_F , then $\omega = (-n + 2k)\omega_F$. The maximal k is n . Therefore the possible frequency components of $Y_n(j\omega)$ are

$$\Omega_n = \{(-n + 2k)\omega_F, k = 0, 1, \dots, n\} \quad (12)$$

Moreover, it is easy to deduce that

$$\Omega = \bigcup_{n=1}^N \Omega_n = \{k\omega_F, k = -N, \dots, -1, 0, 1, \dots, N\} \quad (13)$$

Equation (13) explains why superharmonic components are generated when a nonlinear system is subjected to a harmonic excitation. In the following, only those components with positive frequencies will be considered.

The NOFRFs defined in equation (4) can be extended to the case of harmonic inputs as

$$G_n^H(j\omega) = \frac{\frac{1}{2^n} \sum_{\omega_{k_1} + \dots + \omega_{k_n} = \omega} H_n(j\omega_{k_1}, \dots, j\omega_{k_n}) A(j\omega_{k_1}) \dots A(j\omega_{k_n})}{\frac{1}{2^n} \sum_{\omega_{k_1} + \dots + \omega_{k_n} = \omega} A(j\omega_{k_1}) \dots A(j\omega_{k_n})} \quad n = 1, \dots, N \quad (14)$$

under the condition that

$$A_n(j\omega) = \frac{1}{2^n} \sum_{\omega_{k_1} + \dots + \omega_{k_n} = \omega} A(j\omega_{k_1}) \dots A(j\omega_{k_n}) \neq 0 \quad (15)$$

Obviously, $G_n^H(j\omega)$ is only valid over Ω_n defined by equation (12). Consequently, the output spectrum $Y(j\omega)$ of nonlinear systems under a harmonic input can be expressed as

$$Y(j\omega) = \sum_{n=1}^N Y_n(j\omega) = \sum_{n=1}^N G_n^H(j\omega) A_n(j\omega) \quad (16)$$

When k of the n frequencies of $\omega_{k_1}, \dots, \omega_{k_n}$ are taken as ω_F and the remainders are as $-\omega_F$, substituting equation (9) into equation (15) yields,

$$A_n(j(-n + 2k)\omega_F) = \frac{1}{2^n} |A|^n e^{j(-n+2k)\beta} \quad (17)$$

Thus $G_n^H(j\omega)$ becomes

$$\begin{aligned} G_n^H(j(-n + 2k)\omega_F) &= \frac{\frac{1}{2^n} H_n(\overbrace{j\omega_F, \dots, j\omega_F}^k, \overbrace{-j\omega_F, \dots, -j\omega_F}^{n-k}) |A|^n e^{j(-n+2k)\beta}}{\frac{1}{2^n} |A|^n e^{j(-n+2k)\beta}} \\ &= H_n(\overbrace{j\omega_F, \dots, j\omega_F}^k, \overbrace{-j\omega_F, \dots, -j\omega_F}^{n-k}) \end{aligned} \quad (18)$$

where $H_n(j\omega_1, \dots, j\omega_n)$ is assumed to be a symmetric function. Therefore, in this case, $G_n^H(j\omega)$ over the n th order output frequency range $\Omega_n = \{(-n+2k)\omega_F, k=0,1, \dots, n\}$ is equal to the GFRF $H_n(j\omega_1, \dots, j\omega_n)$ evaluated at $\omega_1 = \dots = \omega_k = \omega_F, \omega_{k+1} = \dots = \omega_n = -\omega_F, k=0, \dots, n$.

3. Analysis of Nonlinear MDOF Systems Using the NOFRFs

3.1 Locally Nonlinear MDOF System

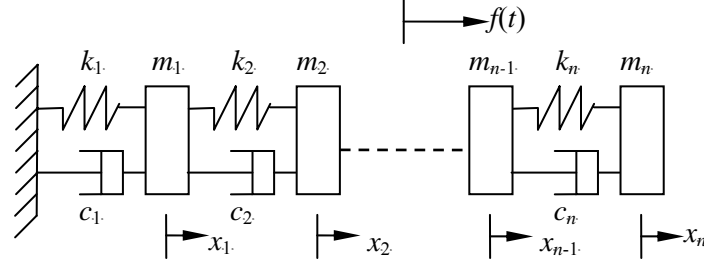


Figure 1, a multi-degree freedom oscillator

The considered multi-degree-of-freedom oscillator is shown as Figure 1, the input force is added on the J th mass.

If all springs and damping are linear, then the governing motion equation of the MDOF oscillator in Figure 1 can be written as

$$M\ddot{x} + C\dot{x} + Kx = F(t) \quad (19)$$

where

$$M = \begin{bmatrix} m_1 & 0 & \dots & 0 \\ 0 & m_2 & \dots & 0 \\ \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & \dots & m_n \end{bmatrix}$$

is the system mass matrix, and

$$C = \begin{bmatrix} c_1 + c_2 & -c_2 & 0 & \dots & 0 \\ -c_2 & c_2 + c_3 & -c_3 & \ddots & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & \ddots & -c_{n-1} & c_{n-1} + c_n & -c_n \\ 0 & \dots & 0 & -c_n & c_n \end{bmatrix} \quad K = \begin{bmatrix} k_1 + k_2 & -k_2 & 0 & \dots & 0 \\ -k_2 & k_2 + k_3 & -k_3 & \ddots & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & \ddots & -k_{n-1} & k_{n-1} + k_n & -k_n \\ 0 & \dots & 0 & -k_n & k_n \end{bmatrix}$$

are the system damping and stiffness matrix respectively. $x = (x_1, \dots, x_n)'$ is the displacement vector, and

$$F(t) = (\underbrace{0, \dots, 0}_{J-1}, f(t), \underbrace{0, \dots, 0}_{n-J})'$$

is the external force vector acting on the oscillator.

Equation (19) is the basis of the modal analysis method, which is a well-established approach for determining dynamic characteristics of engineering structures. In the linear case, the displacements $x_i(t)$ ($i=1, \dots, n$) can be expressed as

$$x_i(t) = \int_{-\infty}^{+\infty} h_{(i)}(t-\tau) f(\tau) d\tau \quad (20)$$

where $h_{(i)}(t)$ ($i=1, \dots, n$) are the impulse response functions that are determined by equation (19), and the Fourier transform of $h_{(i)}(t)$ is the well-known FRF.

Assuming the component between the L th and $(L-1)$ th masses has a nonlinear stiffness and damping, and the restoring forces $S_{LS}(\Delta)$ and $S_{LD}(\dot{\Delta})$ are the polynomial functions of the deformation Δ and its derivative $\dot{\Delta}$ respectively, i.e.,

$$S_{LS}(\Delta) = \sum_{i=1}^P r_i \Delta^i, \quad S_{LD}(\dot{\Delta}) = \sum_{i=1}^P w_i \dot{\Delta}^i \quad (21)$$

where P is the degree of the polynomial. Without loss of generality, assume $L \neq 1, n$ and $L < J$, $k_L = r_1$ and $c_L = w_1$. Then the motion of the oscillator in Figure 1 is determined by equations (22)~(28) in the following.

For the masses that are not connected to the L th spring, the governing motion equations are

$$m_1 \ddot{x}_1 + (c_1 + c_2) \dot{x}_1 - c_2 \dot{x}_2 + (k_1 + k_2) x_1 - k_2 x_2 = 0 \quad (22)$$

$$m_i \ddot{x}_i + (c_i + c_{i+1}) \dot{x}_i - c_i \dot{x}_{i-1} - c_{i+1} \dot{x}_{i+1} + (k_i + k_{i+1}) x_i - k_i x_{i-1} - k_{i+1} x_{i+1} = 0 \quad (i \neq L-1, L, J) \quad (23)$$

$$m_J \ddot{x}_J + (c_J + c_{J+1}) \dot{x}_J - c_J \dot{x}_{J-1} - c_{J+1} \dot{x}_{J+1} + (k_J + k_{J+1}) x_J - k_J x_{J-1} - k_{J+1} x_{J+1} = f(t) \quad (24)$$

$$m_n \ddot{x}_n + c_n \dot{x}_n - c_n \dot{x}_{n-1} + k_n x_n - k_n x_{n-1} = 0 \quad (25)$$

For the mass that is connected to the left of the L th spring, the governing motion equation is

$$m_{L-1} \ddot{x}_{L-1} + (k_{L-1} + k_L) x_{L-1} - k_{L-1} x_{L-2} - k_L x_L + (c_{L-1} + c_L) \dot{x}_{L-1} - c_{L-1} \dot{x}_{L-2} - c_L \dot{x}_L + \sum_{i=2}^P r_i (x_{L-1} - x_L)^i + \sum_{i=2}^P w_i (\dot{x}_{L-1} - \dot{x}_L)^i = 0 \quad (26)$$

For the mass that is connected to the right of the L th spring, the governing motion equation is

$$m_L \ddot{x}_L + (k_L + k_{L+1}) x_L - k_L x_{L-1} - k_{L+1} x_{L+1} + (c_L + c_{L+1}) \dot{x}_L - c_L \dot{x}_{L-1} - c_{L+1} \dot{x}_{L+1} - \sum_{i=2}^P r_i (x_{L-1} - x_L)^i - \sum_{i=2}^P w_i (\dot{x}_{L-1} - \dot{x}_L)^i = 0 \quad (27)$$

Denote

$$NonF = \sum_{i=2}^P w_i (\dot{x}_{L-1} - \dot{x}_L)^i + \sum_{i=2}^P r_i (x_{L-1} - x_L)^i \quad (28)$$

$$NF = \begin{pmatrix} 0 & \dots & 0 & NonF & -NonF & 0 & \dots & 0 \end{pmatrix} \quad (29)$$

Then, equation (22)~(27) can be rewritten in a matrix form as

$$M\ddot{x} + C\dot{x} + Kx = -NF + F(t) \quad (30)$$

The system described by equations (28)~(30) is a typical locally nonlinear MDOF system. The L th nonlinear component can lead the whole system to behave nonlinearly. In this case, the Volterra series can be used to describe the relationships between the displacements $x_i(t)$ ($i=1, \dots, n$) and the input force $f(t)$ as below

$$x_i(t) = \sum_{j=1}^N \int_{-\infty}^{\infty} \dots \int_{-\infty}^{\infty} h_{(i,j)}(\tau_1, \dots, \tau_j) \prod_{i=1}^j f(t - \tau_i) d\tau_i \quad (31)$$

under quite general conditions [3]. In equation (31), $h_{(i,j)}(\tau_1, \dots, \tau_j)$, ($i=1, \dots, n$, $j=1, \dots, N$), represents the j th order Volterra kernel for the relationship between $f(t)$ and the displacement of m_i . The Fourier Transform of $h_{(i,j)}(\tau_1, \dots, \tau_j)$ is the corresponding GFRF $H_{(i,j)}(j\omega_1, \dots, j\omega_j)$ ($i=1, \dots, n$, $j=1, \dots, N$).

3.2 GFRFs of the Locally Nonlinear MDOF System

From equations (22)~(27), the GFRFs $H_{(i,j)}(j\omega_1, \dots, j\omega_j)$, ($i=1, \dots, n$, $j=1, \dots, N$) can be determined using the harmonic probing method [5][6].

First consider the input $f(t)$ is of a single harmonic

$$f(t) = e^{j\omega t} \quad (32)$$

Substituting (30) and

$$x_i(t) = H_{(i,1)}(j\omega) e^{j\omega t} \quad (i=1, \dots, n) \quad (33)$$

into equations (22)~(27) and extracting the coefficients of $e^{j\omega t}$ yields, for the first and n th masses,

$$(-m_1\omega^2 + j(c_1 + c_2)\omega + (k_1 + k_2))H_{(1,1)}(j\omega) - (jc_2\omega + k_2)H_{(2,1)}(j\omega) = 0 \quad (34)$$

$$(-m_n\omega^2 + jc_n\omega + k_n)H_{(n,1)}(j\omega) - (jc_n\omega + k_n)H_{(n-1,1)}(j\omega) = 0 \quad (35)$$

for other masses excluding the J th mass

$$\begin{aligned} &(-m_i\omega^2 + j(c_i + c_{i+1})\omega + k_i + k_{i+1})H_{(i,1)}(j\omega) - (jc_i\omega + k_i)H_{(i-1,1)}(j\omega) \\ &- (jc_{i+1}\omega + k_{i+1})H_{(i+1,1)}(j\omega) = 0 \end{aligned} \quad (i \neq 1, J, n) \quad (36)$$

for the J th mass

$$\begin{aligned} &(-m_J\omega^2 + j(c_J + c_{J+1})\omega + k_J + k_{J+1})H_{(J,1)}(j\omega) - (jc_J\omega + k_J)H_{(J-1,1)}(j\omega) \\ &- (jc_{J+1}\omega + k_{J+1})H_{(J+1,1)}(j\omega) = 1 \end{aligned} \quad (37)$$

Equations (34)~(37) can be written in a matrix form as

$$\left(-M\omega^2 + jC\omega + K\right)H_1(j\omega) = \begin{pmatrix} \overbrace{0 \dots 0}^{J-1} & 1 & \overbrace{0 \dots 0}^{n-J} \end{pmatrix}^T \quad (38)$$

where

$$H_1(j\omega) = \left(H_{(1,1)}(j\omega) \quad \dots \quad H_{(n,1)}(j\omega)\right)^T \quad (39)$$

From equation (39), it is known that

$$H_1(j\omega) = \left(-M\omega^2 + jC\omega + K\right)^{-1} \begin{pmatrix} \overbrace{0 \dots 0}^{J-1} & 1 & \overbrace{0 \dots 0}^{n-J} \end{pmatrix}^T \quad (40)$$

Denote

$$\Theta(j\omega) = -M\omega^2 + jC\omega + K \quad (41)$$

and

$$\Theta^{-1}(j\omega) = \begin{pmatrix} Q_{(1,1)}(j\omega) & \cdots & Q_{(1,n)}(j\omega) \\ \vdots & \ddots & \vdots \\ Q_{(n,1)}(j\omega) & \cdots & Q_{(n,n)}(j\omega) \end{pmatrix} \quad (42)$$

It is obtained from equations (40)~(42) that

$$H_{(i,1)}(j\omega) = Q_{(i,J)}(j\omega) \quad (i = 1, \dots, n) \quad (43)$$

Thus, for any two consecutive masses, the relationship between the first order GFRFs can be expressed as

$$\frac{H_{(i,1)}(j\omega)}{H_{(i+1,1)}(j\omega)} = \frac{Q_{(i,J)}(j\omega)}{Q_{(i+1,J)}(j\omega)} = \lambda_1^{i,i+1}(\omega) \quad (i = 1, \dots, n-1) \quad (44)$$

The above procedure used to analyze the relationships between the first order GFRFs can be extended to investigate the relationship between the \bar{N} th order GFRFs with $\bar{N} \geq 2$. To achieve this, consider the input

$$f(t) = \sum_{k=1}^{\bar{N}} e^{j\omega_k t} \quad (45)$$

Substituting (45) and

$$x_i(t) = H_{(i,1)}(j\omega_1)e^{j\omega_1 t} + \cdots + H_{(i,1)}(j\omega_{\bar{N}})e^{j\omega_{\bar{N}} t} + \cdots \\ + \bar{N}! H_{(i,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})e^{j(\omega_1 + \cdots + \omega_{\bar{N}})t} + \cdots \quad (i = 1, \dots, n) \quad (46)$$

into equations (22)~(27) and extracting the coefficients of $e^{j(\omega_1 + \cdots + \omega_{\bar{N}})t}$ yield

$$\begin{aligned} & \left(-m_1(\omega_1 + \cdots + \omega_{\bar{N}})^2 + j(c_1 + c_2)(\omega_1 + \cdots + \omega_{\bar{N}}) + (k_1 + k_2) \right) H_{(1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_2(\omega_1 + \cdots + \omega_{\bar{N}}) + k_2 \right) H_{(2,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) = 0 \end{aligned} \quad (47)$$

$$\begin{aligned} & \left(-m_n(\omega_1 + \cdots + \omega_{\bar{N}})^2 + jc_n(\omega_1 + \cdots + \omega_{\bar{N}}) + k_n \right) H_{(n,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_n(\omega_1 + \cdots + \omega_{\bar{N}}) + k_n \right) H_{(n-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) = 0 \end{aligned} \quad (48)$$

$$\begin{aligned} & \left(-m_i(\omega_1 + \cdots + \omega_{\bar{N}})^2 + j(c_i + c_{i+1})(\omega_1 + \cdots + \omega_{\bar{N}}) + k_i + k_{i+1} \right) H_{(i,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_i(\omega_1 + \cdots + \omega_{\bar{N}}) + k_i \right) H_{(i-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_{i+1}(\omega_1 + \cdots + \omega_{\bar{N}}) + k_{i+1} \right) H_{(i+1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) = 0 \quad (i \neq 1, L-1, L, n) \end{aligned} \quad (49)$$

$$\begin{aligned} & \left(-m_{L-1}(\omega_1 + \cdots + \omega_{\bar{N}})^2 + j(c_{L-1} + c_L)(\omega_1 + \cdots + \omega_{\bar{N}}) + k_{L-1} + k_L \right) H_{(L-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_{L-1}(\omega_1 + \cdots + \omega_{\bar{N}}) + k_{L-1} \right) H_{(L-2,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_L(\omega_1 + \cdots + \omega_{\bar{N}}) + k_L \right) H_{(L,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) + \Lambda_{\bar{N}}^{L-1,L}(j\omega_1, \dots, j\omega_{\bar{N}}) = 0 \quad (50) \\ & \left(-m_L(\omega_1 + \cdots + \omega_{\bar{N}})^2 + j(c_L + c_{L+1})(\omega_1 + \cdots + \omega_{\bar{N}}) + k_L + k_{L+1} \right) H_{(L,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ & - \left(jc_L(\omega_1 + \cdots + \omega_{\bar{N}}) + k_L \right) H_{(L-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \end{aligned}$$

$$-(jc_{L+1}(\omega_1 + \dots + \omega_{\bar{N}}) + k_{L+1})H_{(L+1, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) - \Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}}) = 0 \quad (51)$$

In equations (50) and (51), $\Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}})$ represents the extra terms introduced by

$NonF = \sum_{i=2}^P w_i (\dot{x}_{L-1} - \dot{x}_L)^i + \sum_{i=2}^P r_i (x_{L-1} - x_L)^i$ for the \bar{N} th order GFRFs, for example, for the second order GFRFs,

$$\begin{aligned} \Lambda_2^{L-1, L}(j\omega_1, j\omega_2) &= (-w_2\omega_1\omega_2 + r_2)(H_{(L-1,1)}(j\omega_1)H_{(L-1,1)}(j\omega_2) \\ &+ H_{(L,1)}(j\omega_1)H_{(L,1)}(j\omega_2) - H_{(L-1,1)}(j\omega_1)H_{(L,1)}(j\omega_2) - H_{(L-1,1)}(j\omega_2)H_{(L,1)}(j\omega_1)) \end{aligned} \quad (52)$$

Denote

$$H_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) = \left(H_{(1, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \quad \dots \quad H_{(n, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \right)^T \quad (53)$$

and

$$\Lambda_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) = \begin{bmatrix} \overbrace{0 \dots 0}^{L-2} & \Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}}) & -\Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}}) & \overbrace{0 \dots 0}^{n-L} \end{bmatrix} \quad (54)$$

then equations (47)~(51) can be written in a matrix form as

$$\Theta(j(\omega_1 + \dots + \omega_{\bar{N}}))H_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) = \Lambda_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) \quad (55)$$

so that

$$H_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) = \Theta^{-1}(j(\omega_1 + \dots + \omega_{\bar{N}}))\Lambda_{\bar{N}}(j\omega_1, \dots, j\omega_{\bar{N}}) \quad (56)$$

Therefore, for each mass, the \bar{N} th order GFRF can be calculated as

$$\begin{aligned} &H_{(i, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ &= \left(Q_{i, L-1}(j(\omega_1 + \dots + \omega_{\bar{N}})), Q_{i, L}(j(\omega_1 + \dots + \omega_{\bar{N}})) \right) \begin{pmatrix} \Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}}) \\ -\Lambda_{\bar{N}}^{L-1, L}(j\omega_1, \dots, j\omega_{\bar{N}}) \end{pmatrix} \\ &\quad (i=1, \dots, n) \end{aligned} \quad (57)$$

Consequently, for two consecutive masses, the \bar{N} th order GFRFs have the following relationships

$$\begin{aligned} \frac{H_{(i, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})}{H_{(i+1, \bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})} &= \frac{Q_{i, L-1}(j(\omega_1 + \dots + \omega_{\bar{N}})) - Q_{i, L}(j(\omega_1 + \dots + \omega_{\bar{N}}))}{Q_{i+1, L-1}(j(\omega_1 + \dots + \omega_{\bar{N}})) - Q_{i+1, L}(j(\omega_1 + \dots + \omega_{\bar{N}}))} \\ &= Q^{i, i+1}(j(\omega_1 + \dots + \omega_{\bar{N}})) = \lambda_{\bar{N}}^{i, i+1}(\omega_1 + \dots + \omega_{\bar{N}}) \quad (i=1, \dots, n-1) \end{aligned} \quad (58)$$

Equations (44) and (58) give a comprehensive description for the relationships between the GFRFs of any two consecutive masses for the nonlinear MDOF system (30).

In addition, denote $\lambda_{\bar{N}}^{0,1}(j\omega_1 + \dots + j\omega_{\bar{N}}) = 0$, ($\bar{N} = 1, \dots, N$), then for the first two masses, from equations (34) and (47), it can be known that

$$\begin{aligned}
& \lambda_{\bar{N}}^{1,2}(\omega_1 + \dots + \omega_{\bar{N}}) \\
&= \frac{H_{(1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})}{H_{(2,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})} = \frac{jc_2(\omega_1 + \dots + \omega_{\bar{N}}) + k_2}{\left[\frac{j((1 - \lambda_{\bar{N}}^{0,1}(\omega_1 + \dots + \omega_{\bar{N}}))c_1 + c_2)(\omega_1 + \dots + \omega_{\bar{N}})}{+ (1 - \lambda_{\bar{N}}^{0,1}(\omega_1 + \dots + \omega_{\bar{N}}))k_1 + k_2 - m_1(\omega_1 + \dots + \omega_{\bar{N}})^2} \right]} \\
& \quad (\bar{N} = 1, \dots, N) \quad (59)
\end{aligned}$$

Starting with equation (59), and iteratively using equations (36) and (49) from the 1st mass until $i=(L-2)$, it can be deduce that, for the masses on the left of the nonlinear spring excluding the $(L-1)$ th mass, the following relationships exist for the system GFRFs

$$\begin{aligned}
& \lambda_{\bar{N}}^{i,i+1}(\omega_1 + \dots + \omega_{\bar{N}}) \\
&= \frac{H_{(i,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})}{H_{(i+1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})} = \frac{jc_{i+1}(\omega_1 + \dots + \omega_{\bar{N}}) + k_{i+1}}{\left[\frac{j((1 - \lambda_{\bar{N}}^{i-1,i}(\omega_1 + \dots + \omega_{\bar{N}}))c_i + c_{i+1})(\omega_1 + \dots + \omega_{\bar{N}})}{+ (1 - \lambda_{\bar{N}}^{i-1,i}(\omega_1 + \dots + \omega_{\bar{N}}))k_i + k_{i+1} - m_i(\omega_1 + \dots + \omega_{\bar{N}})^2} \right]} \\
& \quad (1 < i \leq L-2, \bar{N} = 1, \dots, N) \quad (60)
\end{aligned}$$

Denote $\lambda_{\bar{N}}^{n+1,n}(\omega_1 + \dots + \omega_{\bar{N}}) = 1$, $(\bar{N} = 1, \dots, N)$, $c_{n+1} = 0$ and $k_{n+1} = 0$. Then, for the last two masses, from equations (35) and (48) it is can be deduced that

$$\begin{aligned}
& \lambda_{\bar{N}}^{n,n-1}(\omega_1 + \dots + \omega_{\bar{N}}) = \frac{H_{(n,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})}{H_{(n-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})} = \frac{1}{\lambda_{\bar{N}}^{n-1,n}(\omega_1 + \dots + \omega_{\bar{N}})} \\
&= \frac{jc_n(\omega_1 + \dots + \omega_{\bar{N}}) + k_n}{\left[\frac{j((1 - \lambda_{\bar{N}}^{n+1,n}(\omega_1 + \dots + \omega_{\bar{N}}))c_{n+1} + c_n)(\omega_1 + \dots + \omega_{\bar{N}})}{+ (1 - \lambda_{\bar{N}}^{n+1,n}(\omega_1 + \dots + \omega_{\bar{N}}))k_{n+1} + k_n - m_n(\omega_1 + \dots + \omega_{\bar{N}})^2} \right]} \quad (\bar{N} = 1, \dots, N) \quad (61)
\end{aligned}$$

Starting with equation (61), and iteratively using equations (36) and (49) from n th mass until $i=(J+1)$, it can be deduced that, for the masses on the right of the J th mass, the following relationships can be established for the system GFRFs

$$\begin{aligned}
& \lambda_{\bar{N}}^{i,i-1}(\omega_1 + \dots + \omega_{\bar{N}}) = \frac{H_{(i,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})}{H_{(i-1,\bar{N})}(j\omega_1, \dots, j\omega_{\bar{N}})} = \frac{1}{\lambda_{\bar{N}}^{i-1,i}(\omega_1 + \dots + \omega_{\bar{N}})} \\
&= \frac{jc_i(\omega_1 + \dots + \omega_{\bar{N}}) + k_i}{\left[\frac{j((1 - \lambda_{\bar{N}}^{i+1,i}(\omega_1 + \dots + \omega_{\bar{N}}))c_{i+1} + c_i)(\omega_1 + \dots + \omega_{\bar{N}})}{+ (1 - \lambda_{\bar{N}}^{i+1,i}(\omega_1 + \dots + \omega_{\bar{N}}))k_{i+1} + k_i - m_i(\omega_1 + \dots + \omega_{\bar{N}})^2} \right]} \\
& \quad (J+1 \leq i \leq n, \bar{N} = 1, \dots, N) \quad (62)
\end{aligned}$$

For the masses between the $(L-1)$ th and J th masses, $(L \leq i \leq J)$, the relationships between the GFRFs can also be described as equation (62), but a little modifications are required for $\lambda_1^{J,J-1}(\omega)$ and $\lambda_{\bar{N}}^{L,L-1}(\omega_1 + \dots + \omega_{\bar{N}})$, $(\bar{N} = 1, \dots, N)$.

Denote

$$\bar{\lambda}_1^{J,J-1}(\omega) = \frac{jc_J\omega + k_J}{\left[-m_J\omega^2 + j\left((1 - \lambda_1^{J+1,J}(\omega))c_{J+1} + c_J \right)\omega + (1 - \lambda_1^{J+1,J}(\omega))k_{J+1} + k_J \right]} \quad (63)$$

Then, from equation (37), it can be known that, when $\bar{N} = 1$ and $i = J$, the relationship given in (62) needs to be modified as

$$\lambda_1^{J,J-1}(\omega) = \frac{H_{(J,1)}(j\omega)}{H_{(J-1,1)}(j\omega)} = \frac{1}{\lambda_1^{J-1,J}(\omega)} = \bar{\lambda}_1^{J,J-1}(\omega) \left(1 + \frac{1}{jc_J\omega + k_J} \frac{1}{H_{(J-1,1)}(j\omega)} \right) \quad (64)$$

Denote

$$\bar{\lambda}_N^{L,L-1}(\omega_1 + \dots + \omega_N) = \frac{jc_L(\omega_1 + \dots + \omega_N) + k_L}{\left[j\left((1 - \lambda_N^{L+1,L}(\omega_1 + \dots + \omega_N))c_{L+1} + c_L \right)(\omega_1 + \dots + \omega_N) + (1 - \lambda_N^{L+1,L}(\omega_1 + \dots + \omega_N))k_{L+1} + k_L - m_L(\omega_1 + \dots + \omega_N)^2 \right]} \quad (65)$$

Then, for the L th mass, using equation (51), it can be known that, when $\bar{N} \geq 2$, the relationships given in (62) need to be modified as

$$\begin{aligned} \lambda_N^{L,L-1}(\omega_1 + \dots + \omega_N) &= \frac{H_{(L,\bar{N})}(j\omega_1, \dots, j\omega_N)}{H_{(L-1,\bar{N})}(j\omega_1, \dots, j\omega_N)} = \frac{1}{\lambda_N^{L-1,L}(\omega_1 + \dots + \omega_N)} \\ &= \bar{\lambda}_N^{L,L-1}(\omega_1 + \dots + \omega_N) \left(1 + \frac{1}{jc_L(\omega_1 + \dots + \omega_N) + k_L} \frac{\Lambda_N^{L-1,L}(j\omega_1, \dots, j\omega_N)}{H_{(L-1,\bar{N})}(j\omega_1, \dots, j\omega_N)} \right) \end{aligned} \quad (\bar{N} \geq 2) \quad (66)$$

Under other occasions, if $i = J$, $\bar{N} \neq 1$, and if $i = L$, $\bar{N} = 1$, for the masses ($L \leq i \leq J$), the relationships between the GFRFs can be expressed as

$$\begin{aligned} \lambda_N^{i,i-1}(\omega_1 + \dots + \omega_N) &= \frac{H_{(i,\bar{N})}(j\omega_1, \dots, j\omega_N)}{H_{(i-1,\bar{N})}(j\omega_1, \dots, j\omega_N)} = \frac{1}{\lambda_N^{i-1,i}(\omega_1 + \dots + \omega_N)} \\ &= \frac{jc_i(\omega_1 + \dots + \omega_N) + k_i}{\left[j\left((1 - \lambda_N^{i+1,i}(\omega_1 + \dots + \omega_N))c_{i+1} + c_i \right)(\omega_1 + \dots + \omega_N) + (1 - \lambda_N^{i+1,i}(\omega_1 + \dots + \omega_N))k_{i+1} + k_i - m_i(\omega_1 + \dots + \omega_N)^2 \right]} \\ &\quad (L \leq i \leq J, \bar{N} = 1, \dots, N, \text{ and if } i = J, \bar{N} \neq 1, \text{ and if } i = L, \bar{N} = 1) \end{aligned} \quad (67)$$

From a different respective, equations (59)~(67) give a comprehensive description for the relationships between the GFRFs of any two consecutive masses for the nonlinear MDOF system (30).

3.3 NOFRFs of the Locally Nonlinear MDOF System

According to the definition of NOFRF in equation (4), the \bar{N} th order NOFRF of the i th mass can be expressed as

$$G_{(i,\bar{N})}(j\omega) = \frac{\int_{\omega_1+\dots+\omega_{\bar{N}}=\omega} H_{(i,\bar{N})}(j\omega_1,\dots,j\omega_{\bar{N}}) \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}}{\int_{\omega_1+\dots+\omega_{\bar{N}}=\omega} \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}} \quad (1 \leq \bar{N} \leq N, 1 \leq i \leq n) \quad (68)$$

where $F(j\omega)$ is the Fourier transform of $f(t)$.

According to equation (58), for any $\bar{N} \geq 2$, equation (68) can be rewritten as

$$\begin{aligned} G_{(i,\bar{N})}(j\omega) &= \frac{\int_{\omega_1+\dots+\omega_{\bar{N}}=\omega} Q^{i,i+1}(j(\omega_1+\dots+\omega_{\bar{N}})) H_{(i+1,\bar{N})}(j\omega_1,\dots,j\omega_{\bar{N}}) \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}}{\int_{\omega_1+\dots+\omega_{\bar{N}}=\omega} \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}} \\ &= Q^{i,i+1}(j\omega) G_{(i+1,\bar{N})}(j\omega) \quad (2 \leq \bar{N} \leq N, 1 \leq i \leq n-1) \quad (69) \end{aligned}$$

Then for two consecutive masses, the NOFRFs have the following relationships

$$\frac{G_{(i,\bar{N})}(j\omega)}{G_{(i+1,\bar{N})}(j\omega)} = Q^{i,i+1}(j\omega) = \lambda_{\bar{N}}^{i,i+1}(\omega) \quad (2 \leq \bar{N} \leq N, 1 \leq i \leq n-1) \quad (70)$$

Similarly, according to equation (44), for $\bar{N} = 1$, equation (68) can be rewritten as

$$G_{(i,1)}(j\omega) = \lambda_1^{i,i+1}(\omega) G_{(i+1,1)}(j\omega) \quad (1 \leq i \leq n-1) \quad (71)$$

therefore

$$\frac{G_{(i,1)}(j\omega)}{G_{(i+1,1)}(j\omega)} = \frac{Q_{(i,J)}(j\omega)}{Q_{(i+1,J)}(j\omega)} = \lambda_1^{i,i+1}(\omega) \quad (1 \leq i \leq n-1) \quad (72)$$

Equations (70) and (72) give a comprehensive description for the relationships between the NOFRFs of two consecutive masses of the nonlinear MDOF system (30).

The relationships between the NOFRFs of two consecutive masses can also be derived from equations (59)~(67).

From equation (59), it can be known that

$$\frac{G_{(1,\bar{N})}(j\omega)}{G_{(2,\bar{N})}(j\omega)} = \frac{jc_2\omega + k_2}{(-m_1\omega^2 + (1 - \lambda_{\bar{N}}^{0,1}(\omega))(k_1 + jc_1\omega) + k_2 + jc_2\omega)} = \lambda_{\bar{N}}^{1,2}(\omega) \quad (1 \leq \bar{N} \leq N) \quad (73)$$

Starting with equation (73), and iteratively using equations (60) and (70) from the 1st mass until $i=(L-2)$, it can be deduced that, for the masses on the left of the nonlinear spring excluding the $(L-1)$ th mass, the following relationships exists for the NOFRFs.

$$\lambda_{\bar{N}}^{i,i+1}(j\omega) = \frac{G_{(i,\bar{N})}(j\omega)}{G_{(i+1,\bar{N})}(j\omega)} = \frac{jc_{i+1}\omega + k_{i+1}}{[-m_i\omega^2 + (1 - \lambda_{\bar{N}}^{i-1,i}(j\omega))(jc_i\omega + k_i) + jc_{i+1}\omega + k_{i+1}]} \quad (1 \leq i \leq L-2, \bar{N} = 1, \dots, N) \quad (74)$$

Similarly, for the masses on at the right of the J th mass, the following relationship about the NOFRFs can be established using equations (62) and (68).

$$\lambda_{\bar{N}}^{i,i-1}(j\omega) = \frac{1}{\lambda_{\bar{N}}^{i-1,i}(j\omega)} = \frac{G_{(i,\bar{N})}(j\omega)}{G_{(i-1,\bar{N})}(j\omega)} = \frac{jc_i\omega + k_i}{[-m_i\omega^2 + (1 - \lambda_{\bar{N}}^{i+1,i}(j\omega))(jc_{i+1}\omega + k_{i+1}) + jc_i\omega + k_i]} \quad (J+1 \leq i \leq n, \bar{N} = 1, \dots, N) \quad (75)$$

Starting from the J th mass, it is easily deduced that the NOFRFs of two consecutive masses, which locate between the $(L-1)$ th and J th masses, have the similar relationships given in equation (75), but a little modifications are required for $\lambda_1^{J,J-1}(\omega)$ and $\lambda_{\bar{N}}^{L,L-1}(\omega)$, ($\bar{N} = 1, \dots, N$).

From equation (64), it can be known that, when $\bar{N} = 1$ and $i = J$, the relationship given in equation (75) needs to be modified as

$$\lambda_1^{J,J-1}(\omega) = \frac{G_{(J,1)}(j\omega)}{G_{(J-1,1)}(j\omega)} = \bar{\lambda}_1^{J,J-1}(\omega) \left(1 + \frac{1}{jc_J\omega + k_J} \frac{1}{G_{(J-1,1)}(j\omega)} \right) = \frac{1}{\lambda_1^{J-1,J}(\omega)} \quad (76)$$

Denote

$$\bar{\lambda}_{\bar{N}}^{L,L-1}(\omega) = \frac{jc_L\omega + k_L}{[-m_L\omega^2 + (1 - \lambda_{\bar{N}}^{L+1,L}(\omega))(jc_{L+1}\omega + k_{L+1}) + jc_L\omega + k_L]} \quad (77)$$

and

$$\Gamma_{(L-1,\bar{N})}(j\omega) = \frac{\int_{\omega_1 + \dots + \omega_{\bar{N}} = \omega} \Lambda_{\bar{N}}^{L-1,L}(j\omega_1, \dots, j\omega_{\bar{N}}) \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}}{\int_{\omega_1 + \dots + \omega_{\bar{N}} = \omega} \prod_{q=1}^{\bar{N}} F(j\omega_q) d\sigma_{\bar{N}\omega}} \quad (2 \leq \bar{N} \leq N) \quad (78)$$

Then, for the L th mass, from equations (65) and (66), it can be known that, when $\bar{N} \geq 2$, the relationship given in (75) needs to be modified as

$$\lambda_{\bar{N}}^{L,L-1}(j\omega) = \frac{G_{(L,\bar{N})}(j\omega)}{G_{(L-1,\bar{N})}(j\omega)} = \bar{\lambda}_{\bar{N}}^{L,L-1}(j\omega) \left(1 + \frac{1}{jc_L\omega + k_L} \frac{\Gamma_{(L-1,\bar{N})}(j\omega)}{G_{(L-1,\bar{N})}(j\omega)} \right) = \frac{1}{\lambda_{\bar{N}}^{L-1,L}(j\omega)} \quad (2 \leq \bar{N} \leq N) \quad (79)$$

Under other conditions, if $i=J$, $\bar{N} \neq 1$, and if $i=L$, $\bar{N} = 1$, for the masses ($L \leq i \leq J$), from equation (67), it is known that the relationships between the NOFRFs can be expressed as

$$\lambda_{\bar{N}}^{i,i-1}(j\omega) = \frac{1}{\lambda_{\bar{N}}^{i-1,i}(j\omega)} = \frac{G_{(i,\bar{N})}(j\omega)}{G_{(i-1,\bar{N})}(j\omega)} = \frac{jc_i\omega + k_i}{[-m_i\omega^2 + (1 - \lambda_{\bar{N}}^{i+1,i}(j\omega))(jc_{i+1}\omega + k_{i+1}) + jc_i\omega + k_i]} \quad (L \leq i \leq J, \bar{N} = 1, \dots, N, \text{ and if } i=J, \bar{N} \neq 1, \text{ and if } i=L, \bar{N} = 1) \quad (80)$$

From a different perspective, equations (73)~(80) also give a comprehensive description for the relationships between the NFRFs of any two consecutive masses of the nonlinear MDOF system (30).

3.4 The Properties of NOFRFs of Locally Nonlinear Systems

Important properties of the NOFRFs of locally nonlinear MDOF systems can be obtained from equations (70)~(80) as the following

- i) For the masses which are on the left of the nonlinear spring or on the right of the input force, below relationships hold.

$$\frac{G_{(i,1)}(j\omega)}{G_{(i+1,1)}(j\omega)} = \dots = \frac{G_{(i,N)}(j\omega)}{G_{(i+1,N)}(j\omega)} \quad (1 \leq i \leq L-2, J \leq i < n) \quad (81)$$

that is

$$\lambda_1^{i,i+1}(\omega) = \dots = \lambda_N^{i,i+1}(\omega) = \lambda^{i,i+1}(\omega) \quad (1 \leq i \leq L-2, J \leq i < n) \quad (82)$$

- ii) For the masses located between the nonlinear spring and the input force, the following relationships hold.

$$\frac{G_{(i,1)}(j\omega)}{G_{(i+1,1)}(j\omega)} \neq \frac{G_{(i,2)}(j\omega)}{G_{(i+1,2)}(j\omega)} = \dots = \frac{G_{(i,N)}(j\omega)}{G_{(i+1,N)}(j\omega)} \quad (L-1 \leq i \leq J-1) \quad (83)$$

that is

$$\lambda_1^{i,i+1}(\omega) \neq \lambda_2^{i,i+1}(\omega) = \dots = \lambda_N^{i,i+1}(\omega) \quad (L-1 \leq i \leq J-1) \quad (84)$$

- iii) For the masses which are on the left of the nonlinear spring or on the right of the input force, the following relationships about the output frequency responses hold

$$x_i(j\omega) = \lambda^{i,i+1}(j\omega)x_{i+1}(j\omega) \quad (1 \leq i \leq L-2, J \leq i < n) \quad (85)$$

The first property is straightforward. For the masses on the left of the nonlinear spring, from equation (73), it can be known that

$$\lambda_1^{1,2}(j\omega) = \dots = \lambda_N^{1,2}(j\omega) = \frac{k_2 + jc_2\omega}{(-m_1\omega^2 + j\omega(c_1 + c_2) + k_1 + k_2)} = \lambda^{1,2}(j\omega) \quad (86)$$

Consequently, substituting (86) into equation (73) yields

$$\begin{aligned} \lambda_1^{2,3}(\omega) = \dots = \lambda_N^{2,3}(\omega) &= \frac{j c_3 \omega + k_3}{[-m_2 \omega^2 + j((1 - \lambda^{1,2}(j\omega))c_2 + c_3)\omega + (1 - \lambda^{1,2}(j\omega))k_2 + k_3]} \\ &= \lambda^{2,3}(j\omega) \end{aligned} \quad (87)$$

Iteratively use the above procedure until $i=(L-2)$, for the masses ($1 \leq i \leq L-2$) the first property can be proved. Similarly, starting from the n th mass, and iteratively using equation (75) until $i=J$, the first property for the masses ($J \leq i < N$) can also be proved.

From equation (70), it can be known that, for the masses located between the nonlinear spring and the input force, the following relationship is tenable

$$\frac{G_{(i,2)}(j\omega)}{G_{(i+1,2)}(j\omega)} = \dots = \frac{G_{(i,N)}(j\omega)}{G_{(i+1,N)}(j\omega)} = Q^{i,i+1}(j\omega) = \lambda^{i,i+1}(j\omega) = \frac{1}{\lambda^{i+1,i}(j\omega)} \quad (L-1 \leq i \leq J-1) \quad (88)$$

So part of the second property has been proved.

According to Property i), it can be known that

$$\lambda_1^{J+1,J}(\omega) = \frac{1}{\lambda_1^{J,J+1}(\omega)} = \dots = \lambda_N^{J+1,J}(\omega) = \frac{1}{\lambda_N^{J,J+1}(\omega)} \quad (89)$$

Substituting (89) into $\bar{\lambda}_1^{J,J-1}(j\omega)$, $\lambda_2^{J,J-1}(\omega)$, \dots , $\lambda_N^{J,J-1}(\omega)$, it follows

$$\begin{aligned} \bar{\lambda}_1^{J,J-1}(\omega) &= \lambda_2^{J,J-1}(\omega) = \dots = \lambda_N^{J,J-1}(\omega) \\ &= \frac{jc_J\omega + k_J}{\left[-m_J\omega^2 + j\left((1 - \lambda_1^{J+1,J}(\omega))c_{J+1} + c_J\right)\omega + (1 - \lambda_1^{J+1,J}(\omega))k_{J+1} + k_J\right]} \end{aligned} \quad (90)$$

According equation (76), it is known that

$$\lambda_1^{J,J-1}(\omega) = \bar{\lambda}_1^{J,J-1}(\omega) \left(1 + \frac{1}{jc_J\omega + k_J} \frac{1}{G_{(J-1,1)}(j\omega)}\right) \quad (91)$$

As $G_{(J-1,1)}(j\omega) \neq 0$, obviously

$$\lambda_1^{J,J-1}(\omega) \neq \bar{\lambda}_1^{J,J-1}(\omega) = \lambda_2^{J,J-1}(\omega) = \dots = \lambda_N^{J,J-1}(\omega) \quad (92)$$

then

$$\lambda_1^{J-1,J}(\omega) = \frac{1}{\lambda_1^{J,J-1}(\omega)} \neq \lambda_2^{J-1,J}(\omega) = \frac{1}{\lambda_2^{J,J-1}(\omega)} = \dots = \lambda_N^{J-1,J}(\omega) = \frac{1}{\lambda_N^{J,J-1}(\omega)} \quad (93)$$

Substituting (93) into $\lambda_1^{J-1,J-2}(\omega)$, $\lambda_2^{J-1,J-2}(\omega)$, \dots , $\lambda_N^{J-1,J-2}(\omega)$, it can be proved that

$$\lambda_1^{J-2,J-1}(\omega) = \frac{1}{\lambda_1^{J-1,J-2}(\omega)} \neq \lambda_2^{J-2,J-1}(\omega) = \frac{1}{\lambda_2^{J-1,J-2}(\omega)} = \dots = \lambda_N^{J-2,J-1}(\omega) = \frac{1}{\lambda_N^{J-1,J-2}(\omega)} \quad (94)$$

Iteratively using above procedure until $i=(L-1)$, then the property

$$\lambda_1^{i,i+1}(j\omega) \neq \lambda_2^{i,i+1}(j\omega) = \dots = \lambda_N^{i,i+1}(j\omega) \quad (L-1 \leq i \leq J-1)$$

can be proved. By now, the whole second property is proved.

The third property is also straightforward since, according to equation (6), the output frequency response of the i th mass can be expressed as

$$x_{i+1}(j\omega) = \sum_{k=1}^N G_{(i+1,k)}(j\omega) F_k(j\omega) \quad (95)$$

Using the first property, equation (95) can be written as

$$x_{i+1}(j\omega) = \sum_{k=1}^N \lambda^{i,i+1}(j\omega) G_{(i,k)}(j\omega) F_k(j\omega) \quad (96)$$

Obviously, $x_{i+1}(j\omega) = \lambda^{i,i+1}(j\omega)x_i(j\omega)$, then the third property is proved.

The above three properties can be easily extended to more general cases, as follows.

iv) For any two masses which are either on the left of the nonlinear spring or on the right of the input force, the following relationships hold.

$$\begin{aligned} \frac{G_{(i,1)}(j\omega)}{G_{(i+k,1)}(j\omega)} &= \dots = \frac{G_{(i,N)}(j\omega)}{G_{(i+k,N)}(j\omega)} = \lambda^{i,i+k}(\omega) \\ (1 \leq i \leq L-2 \text{ and } i+k \leq L-1 \text{ or } J \leq i < n \text{ and } J < i+k \leq n) \end{aligned} \quad (97)$$

and

$$\lambda^{i,i+k}(\omega) = \prod_{d=0}^{k-1} \lambda^{i+d,i+d+1}(\omega) \quad (98)$$

v) For any two masses which are either on the left of the nonlinear spring or on the right of the input force, the following relationships hold

$$x_i(j\omega) = \lambda^{i,i+k}(\omega)x_{i+k}(j\omega) \\ (1 \leq i \leq L-2 \text{ and } i+k \leq L-1 \text{ or } J \leq i < n \text{ and } J < i+k \leq n) \quad (99)$$

vi) For any two masses located between the nonlinear spring and the input force, the following relationships can be deduced from property ii).

$$\frac{G_{(i,1)}(j\omega)}{G_{(i+k,1)}(j\omega)} \neq \frac{G_{(i,2)}(j\omega)}{G_{(i+k,2)}(j\omega)} = \dots = \frac{G_{(i,N)}(j\omega)}{G_{(i+k,N)}(j\omega)} = \lambda^{i,i+k}(\omega) \\ (L-1 \leq i \leq J-1 \text{ and } L \leq i+k \leq J) \quad (100)$$

and

$$\lambda^{i,i+k}(\omega) = \prod_{d=0}^{k-1} \lambda^{i+d,i+d+1}(\omega) \quad (101)$$

vii) For any two masses at the different sides of the nonlinear spring or at the different sides of the input force, the following relationships hold.

$$\frac{G_{(i,1)}(j\omega)}{G_{(k,1)}(j\omega)} \neq \frac{G_{(i,2)}(j\omega)}{G_{(k,2)}(j\omega)} = \dots = \frac{G_{(i,N)}(j\omega)}{G_{(k,N)}(j\omega)} \\ (1 \leq i \leq L-1 \text{ and } L \leq k \leq n \text{ or } 1 \leq i \leq J-1 \text{ and } J \leq k \leq n) \quad (102)$$

The proof of properties (iv)-(vii) only needs some simple calculations. The details are omitted here.

4 Numerical Study

To verify the analysis results in Section 3, a damped 8-DOF oscillator was used to conduct numerical studies, in which the fourth spring was nonlinear. As widely used in modal analysis, the damping was assumed to be proportional to the damping, e.g., $C = \mu K$. The values of the system parameters are taken as

$$m_1 = \dots = m_8 = 1, \quad r_1 = k_1 = \dots = k_8 = 3.5531 \times 10^4, \quad \mu = 0.01 \\ r_2 = 0.8 \times r_1^2, \quad r_3 = 0.4 \times r_1^3, \quad w_1 = \mu r_1, \quad w_2 = 0.1 \mu^2 k_2, \quad w_3 = 0$$

and the input was a harmonic force acting on the 6th mass, $f(t) = A \sin(2\pi \times 20t)$.

If only the NOFRFs up to the 4th order is considered, according to equations (16) and (17), the frequency components of the outputs of the 8 masses can be written as

$$x_i(j\omega_F) = G_{(i,1)}^H(j\omega_F)F_1(j\omega_F) + G_{(i,3)}^H(j\omega_F)F_3(j\omega_F) \\ x_i(j2\omega_F) = G_{(i,2)}^H(j2\omega_F)F_2(j2\omega_F) + G_{(i,4)}^H(j2\omega_F)F_4(j2\omega_F) \\ x_i(j3\omega_F) = G_{(i,3)}^H(j3\omega_F)F_3(j3\omega_F)$$

$$x_i(j4\omega_F) = G_{(i,4)}^H(j4\omega_F)F_4(j4\omega_F) \quad (i=1, \dots, 8) \quad (103)$$

From equation (103), it can be seen that, using the method in [9], two different inputs with the same waveform but different strengths are sufficient to estimate the NOFRFs up to 4th order. Therefore, in this numerical study, two different inputs are used with $A=0.8$ and $A=1.0$ respectively. The simulation studies were conducted using a fourth-order *Runge–Kutta* method to obtain the forced response of the system.

The evaluated results of $G_1^H(j\omega_F)$, $G_3^H(j\omega_F)$, $G_2^H(j2\omega_F)$ and $G_4^H(j2\omega_F)$ for all masses are given in Table 1 and Table 2. According to the analysis results in the previous section, it is known that the following relationships should be tenable.

$$\begin{aligned} \lambda_1^{i,i+1}(j\omega_F) &= \frac{G_{(i,1)}^H(j\omega_F)}{G_{(i+1,1)}^H(j\omega_F)} = \frac{G_{(i,3)}^H(j\omega_F)}{G_{(i+1,3)}^H(j\omega_F)} = \lambda_3^{i,i+1}(j\omega_F) & \text{for } i=1,2,6,7 \\ \lambda_1^{i,i+1}(j\omega_F) &= \frac{G_{(i,1)}^H(j\omega_F)}{G_{(i+1,1)}^H(j\omega_F)} \neq \frac{G_{(i,3)}^H(j\omega_F)}{G_{(i+1,3)}^H(j\omega_F)} = \lambda_3^{i,i+1}(j\omega_F) & \text{for } i=3,4,5 \\ \lambda_2^{i,i+1}(j2\omega_F) &= \frac{G_{(i,2)}^H(j2\omega_F)}{G_{(i+1,2)}^H(j2\omega_F)} = \frac{G_{(i,4)}^H(j2\omega_F)}{G_{(i+1,4)}^H(j2\omega_F)} = \lambda_4^{i,i+1}(j2\omega_F) & \text{for } i=1, \dots, 7 \end{aligned} \quad (104)$$

Table 1, the evaluated results of $G_1^H(j\omega_F)$ and $G_3^H(j\omega_F)$

	$G_1^H(j\omega_F)(\times 10^{-6})$	$G_3^H(j\omega_F)(\times 10^{-9})$
Mass 1	-1.944241903169+2.877586360412i	5.458634750380-7.366308497467i
Mass 2	-4.176583568969+4.838272030553i	11.57208415372-12.28123597097i
Mass 3	-6.736933696283+5.060776930464i	18.34919935969-12.57357666987i
Mass 4	-9.231909382744+2.952034620849i	-12.79690672083+5.455660873182i
Mass 5	-10.77575870137-1.664260696852i	-5.435201264096+7.592249482842i
Mass 6	-10.10143823260-8.327479816646i	1.220710849413+7.243210130819i
Mass 7	-15.11217068758-0.8377017815558i	6.097443770670+5.910417338742i
Mass 8	-17.33646514165+3.523672640104i	8.643601249962+4.879496926152i

Table 2, the evaluated results of $G_2^H(j2\omega_F)$ and $G_4^H(j2\omega_F)$

	$G_2^H(j2\omega_F)(\times 10^{-9})$	$G_4^H(j2\omega_F)(\times 10^{-10})$
Mass 1	6.021454230962-12.98554218901i	-1.952110843919-3.410834151117i
Mass 2	18.50884595656-19.14114684937i	-1.347439298947-7.185522342534i
Mass 3	38.19859798519-9.325491210626i	3.976662752654-10.03496188850i
Mass 4	-38.08895161488+6.216540013903i	-4.655561901388+9.516667048108i
Mass 5	-16.52707738569+16.85454389201i	1.150036085197+6.378527258423i
Mass 6	-1.252587907918+13.28718243146i	2.777020782522+2.390650901502i
Mass 7	6.213180893960+5.729219962027i	2.269896229546-0.4816873854849i
Mass 8	8.669835294787+0.5734813561479i	1.505312897830-1.850711607214i

Table 3, the evaluated and theoretical values of $\lambda_1^{i,i+1}(j\omega_F)$

	Evaluated	Theoretical
$i=1$	0.539568255325-0.063929850245i	0.539568125657-0.063929279767i
$i=2$	0.741189550115-0.161390493546i	0.741190530388-0.161389040027i
$i=3$	0.821079057389-0.285631391869i	0.821050118371-0.285603877540i
$i=4$	0.795445087122-0.396803857097i	0.795467221982-0.396815218346i
$i=5$	0.715984832936-0.425492731775i	0.715981624553-0.425452611957i
$i=6$	0.696835117313+0.512417438733i	0.696881219804+0.512400603715i
$i=7$	0.827684503605+0.216548817240i	0.827677050338+0.216547405081i

Table 4, the evaluated and theoretical values of $\lambda_3^{i,i+1}(j\omega_F)$

	Evaluated	Theoretical
$i=1$	0.539559368397-0.063934254540i	0.539568125657-0.063929279767i
$i=2$	0.741241865615-0.161378950912i	0.741190530388-0.161389040027i
$i=3$	-1.567808155157 +0.314149907382i	-1.567764971393+0.314134164548i
$i=4$	1.272881818401+0.774281438682i	1.272989592311+0.774409671174i
$i=5$	0.896268022564+0.901435309110i	0.896263983491+0.901393838300i
$i=6$	0.696884187038+0.512400583943i	0.696881219804+0.512400603715i
$i=7$	0.827675930682+0.216550385077i	0.827677050338+0.216547405081i

Table 5, the evaluated and theoretical values of $|\lambda_1^{i,i+1}(j\omega_F)|$ and $|\lambda_3^{i,i+1}(j\omega_F)|$

	$ \lambda_1^{i,i+1}(j\omega_F) $		$ \lambda_3^{i,i+1}(j\omega_F) $	
	Evaluated	Theoretical	Evaluated	Theoretical
$i=1$	0.543342367119	0.543342171230	0.543334060158	0.543342171230
$i=2$	0.758557078015	0.758557726596	0.758605740248	0.758557726596
$i=3$	0.869342343673	0.869305971303	1.598972349882	1.598926914797
$i=4$	0.888924174288	0.888949052960	1.489879146073	1.490037865607
$i=5$	0.832873547298	0.832850293702	1.271173467620	1.271141211600
$i=6$	0.864957115837	0.864984284946	0.864986663810	0.864984284946
$i=7$	0.855543703007	0.855536135008	0.855535806093	0.855536135008

Table 6, the evaluated and theoretical values of $\lambda_2^{i,i+1}(j2\omega_F)$

	Evaluated	Theoretical
$i=1$	0.507797183488-0.176441131870i	0.507802742120-0.176493868719i
$i=2$	0.572740375500-0.361271414124i	0.572987984997-0.361057530431i
$i=3$	-1.015780193448+0.079047935558i	-1.015815300438+0.079073403521i
$i=4$	1.317748969503+0.967716037022i	1.317984293172+0.966962918880i
$i=5$	1.373531878255+1.114352726037i	1.373306980736+1.114467844922i
$i=6$	0.956810202664+1.256265744021i	0.956791143622+1.256208857322i
$i=7$	0.757042356332+0.610746352706i	0.757019558226+0.610676456673i

Table 7, the evaluated and theoretical values of $\lambda_4^{i,i+1}(j2\omega_F)$

	Evaluated	Theoretical
$i=1$	0.507770526862-0.176454935457i	0.507802742120-0.176493868719i
$i=2$	0.572874980296-0.361293838014i	0.572987984997-0.361057530431i
$i=3$	-1.015785489514+0.079060186566i	-1.015815300438+0.079073403521i
$i=4$	1.317558601335+0.967433639033i	1.317984293172+0.966962918880i
$i=5$	1.373533098461+1.114463794374i	1.373306980736+1.114467844922i
$i=6$	0.956829585455+1.256244054577i	0.956791143622+1.256208857322i
$i=7$	0.757036113181+0.610748859977i	0.757019558226+0.610676456673i

Table 8, the evaluated and theoretical values of $|\lambda_2^{i,i+1}(j2\omega_F)|$ and $|\lambda_4^{i,i+1}(j2\omega_F)|$

	$ \lambda_2^{i,i+1}(j2\omega_F) $		$ \lambda_4^{i,i+1}(j2\omega_F) $	
	Evaluated	Theoretical	Evaluated	Theoretical
$i=1$	0.537577392171	0.537599954055	0.537556743234	0.537599954055
$i=2$	0.677162146307	0.677257536859	0.677287959760	0.677257536859
$i=3$	1.018851302947	1.018888280308	1.018857533617	1.018888280308
$i=4$	1.634911824820	1.634655891476	1.634591237553	1.634655891476
$i=5$	1.768720390172	1.768618285754	1.768791316560	1.768618285754
$i=6$	1.579141977000	1.579085173677	1.579136466637	1.579085173677
$i=7$	0.972689177808	0.972627547560	0.972685893093	0.972627547560

From the NOFRFs in Table 1 and Table 2, $\lambda_1^{i,i+1}(j\omega_F)$, $\lambda_3^{i,i+1}(j\omega_F)$, $\lambda_2^{i,i+1}(j2\omega_F)$ and $\lambda_4^{i,i+1}(j2\omega_F)$ ($i=1, \dots, 7$) can be evaluated. Moreover, from equations (44) and (58), the theoretical values of $\lambda_1^{i,i+1}(j\omega_F)$, $\lambda_3^{i,i+1}(j\omega_F)$, $\lambda_2^{i,i+1}(j2\omega_F)$ and $\lambda_4^{i,i+1}(j2\omega_F)$ ($i=1, \dots, 7$) can also be calculated. Both the evaluated and theoretical values of $\lambda_1^{i,i+1}(j\omega_F)$, $\lambda_3^{i,i+1}(j\omega_F)$, $\lambda_2^{i,i+1}(j2\omega_F)$ and $\lambda_4^{i,i+1}(j2\omega_F)$ ($i=1, \dots, 7$) are given in Tables 3, 4, 6 and 7. Their moduli are given in Table 5 and Table 8.

It can be seen that the evaluated results match the theoretical results very well. Moreover, the results shown in Tables 5 and 8 have a strict accordance with the relationships in (104). Therefore, the numerical study verifies the properties of NOFRFs of the locally nonlinear MDOF systems described in Section 3.

From Table 5, it can be seen that $|\lambda_1^{i,i+1}(j\omega_F)|$ and $|\lambda_3^{i,i+1}(j\omega_F)|$ at the 4th and 5th masses are only slight different, but have a significant difference at the 3rd mass. This means that, for the two masses connected to the nonlinear spring, their $|\lambda_1^{i,i+1}(j\omega_F)|$ and $|\lambda_3^{i,i+1}(j\omega_F)|$ have a considerable difference. This result implies that a class of novel approaches can be developed based on the properties of NOFRFs derived in the present study for MDOF nonlinear systems to detect and locate fault elements which make engineering structures behave nonlinearly. This is the focus of our current research studies. The results will be present in a series of later publications.

5 Conclusions

In the present study, the relationships between the NOFRFs of MDOF nonlinear system have been investigated to reveal important properties of nonlinear system. The derivation

considered general cases where the input force was allowed to be added at any mass in the system and the damping characteristics were also taken into account. The results have considerable significance for the application of the NOFRF concept in engineering practices to locate the position of the nonlinear element in a locally nonlinear MDOF system and to diagnose faults in engineering systems which make the system behave nonlinearly. Further research results on this application will be discussed in later publications.

Acknowledgements

The authors gratefully acknowledge the support of the Engineering and Physical Science Research Council, UK, for this work.

References

1. R.K. Pearson, *Discrete Time Dynamic Models*. Oxford University Press, 1994
2. K. Worden, G. Manson, G.R. Tomlinson, A harmonic probing algorithm for the multi-input Volterra series. *Journal of Sound and Vibration* **201**(1997) 67-84
3. Z. Q. Lang, S. A. Billings, Output frequency characteristics of nonlinear system, *International Journal of Control* **64** (1996) 1049-1067.
4. S.A. Billings, K.M. Tsang, Spectral analysis for nonlinear system, part I: parametric non-linear spectral analysis. *Mechanical Systems and Signal Processing*, **3** (1989) 319-339
5. S.A. Billings, J.C. Peyton Jones, Mapping nonlinear integro-differential equations into the frequency domain, *International Journal of Control* **52** (1990) 863-879.
6. J.C. Peyton Jones, S.A. Billings, A recursive algorithm for the computing the frequency response of a class of nonlinear difference equation models. *International Journal of Control* **50** (1989) 1925-1940.
7. H. Zhang, S. A. Billings, Analysing non-linear systems in the frequency domain, I: the transfer function, *Mechanical Systems and Signal Processing* **7** (1993) 531–550.
8. H. Zhang, S. A. Billings, Analysing nonlinear systems in the frequency domain, II: the phase response, *Mechanical Systems and Signal Processing* **8** (1994) 45–62.
9. Z. Q. Lang, S. A. Billings, Energy transfer properties of nonlinear systems in the frequency domain, *International Journal of Control* **78** (2005) 354-362.

10. D.T. Westwick, R.E. Kearney, A new algorithm for identification of multi-input Wiener systems. *Biological Cybernetics*, **68** (1992) 75-85
11. Z.K. Peng, Z.Q. Lang, S.A. Billings, Nonlinear Output Frequency Response Functions for Multi-Input Nonlinear Volterra Systems. (2006), submitted to *International Journal of Control*
12. T.G. Chondros, A.D. Dimarogonas, J. Yao, Vibration of a beam with breathing crack, *Journal of Sound and Vibration*, **239** (2001) 57-67