



44<sup>TH</sup> **TURBOMACHINERY** & 31<sup>ST</sup> **PUMP** SYMPOSIA  
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# Eliminating cavitation damage in high-energy water injection pumps by suction impeller redesign

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# Introduction – Project Objective

## Problem statement:

First-stage/suction impeller of high-energy high-speed water injection pumps (WIP's) is experiencing cavitation damage, resulting in not making the 40,000 hours life criterion.

## Objective:

Design a new first-stage/suction impeller using Computational Fluid Dynamics (CFD) that will minimize (eliminate?) cavitation, thus improving lifetime, and verify this with a flow visualization test.



# Introduction – Pump Info/History

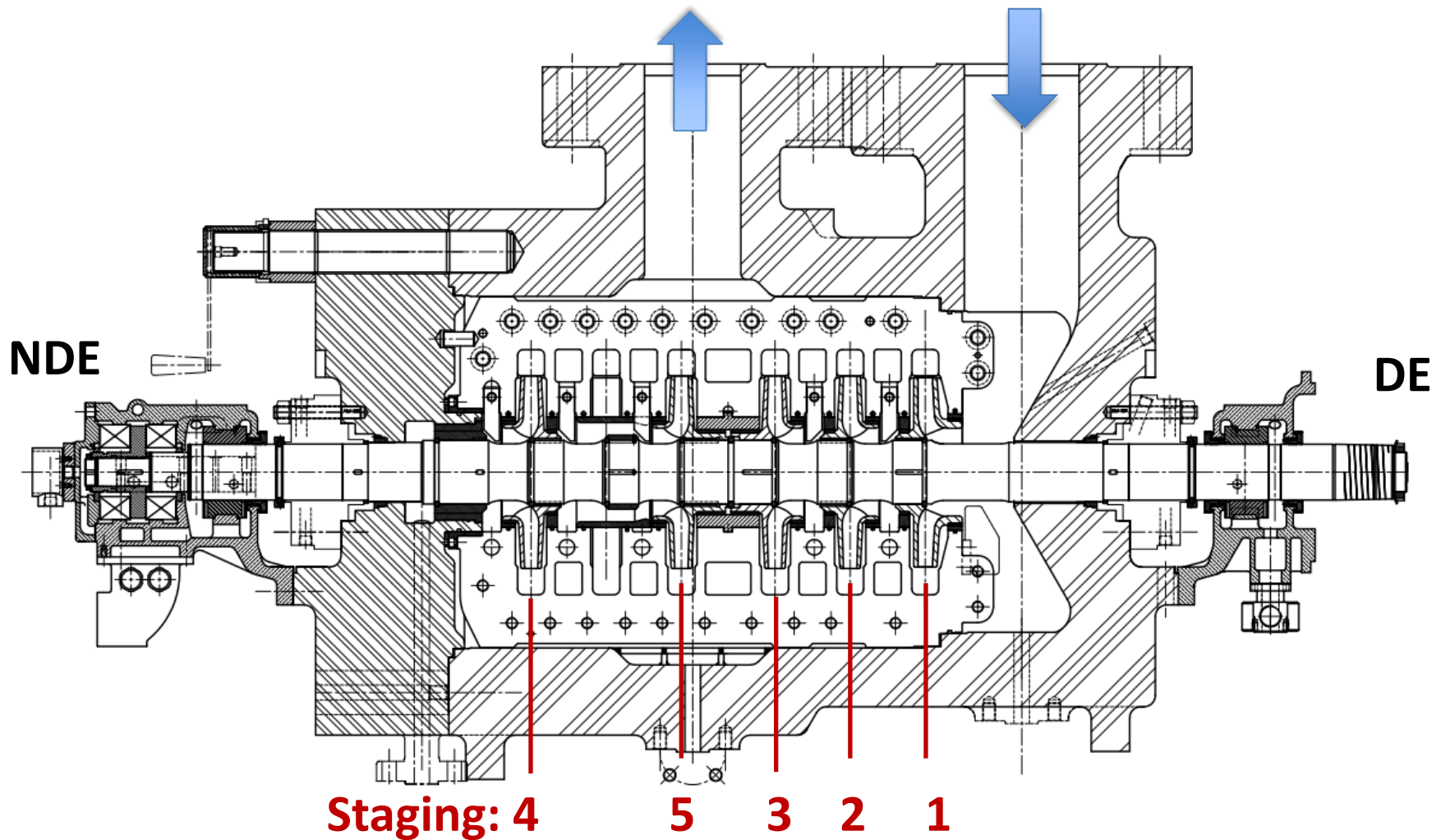
- **Three identical high-speed multistage pumps in water injection service** (Pumps were ordered in 2001 and commissioned in 2005)

## Some Details:

- Five stages (de-staged)
- Single suction first stage
- Between bearing
- Radial split barrel  
(API 610 BB5)
- Top-top nozzles
- High speed (5000 [RPM])
- Fluid: raw (formation) water
- Total Dissolved Solids (TDS): 11000 ppm



# Introduction – Pump Info/History

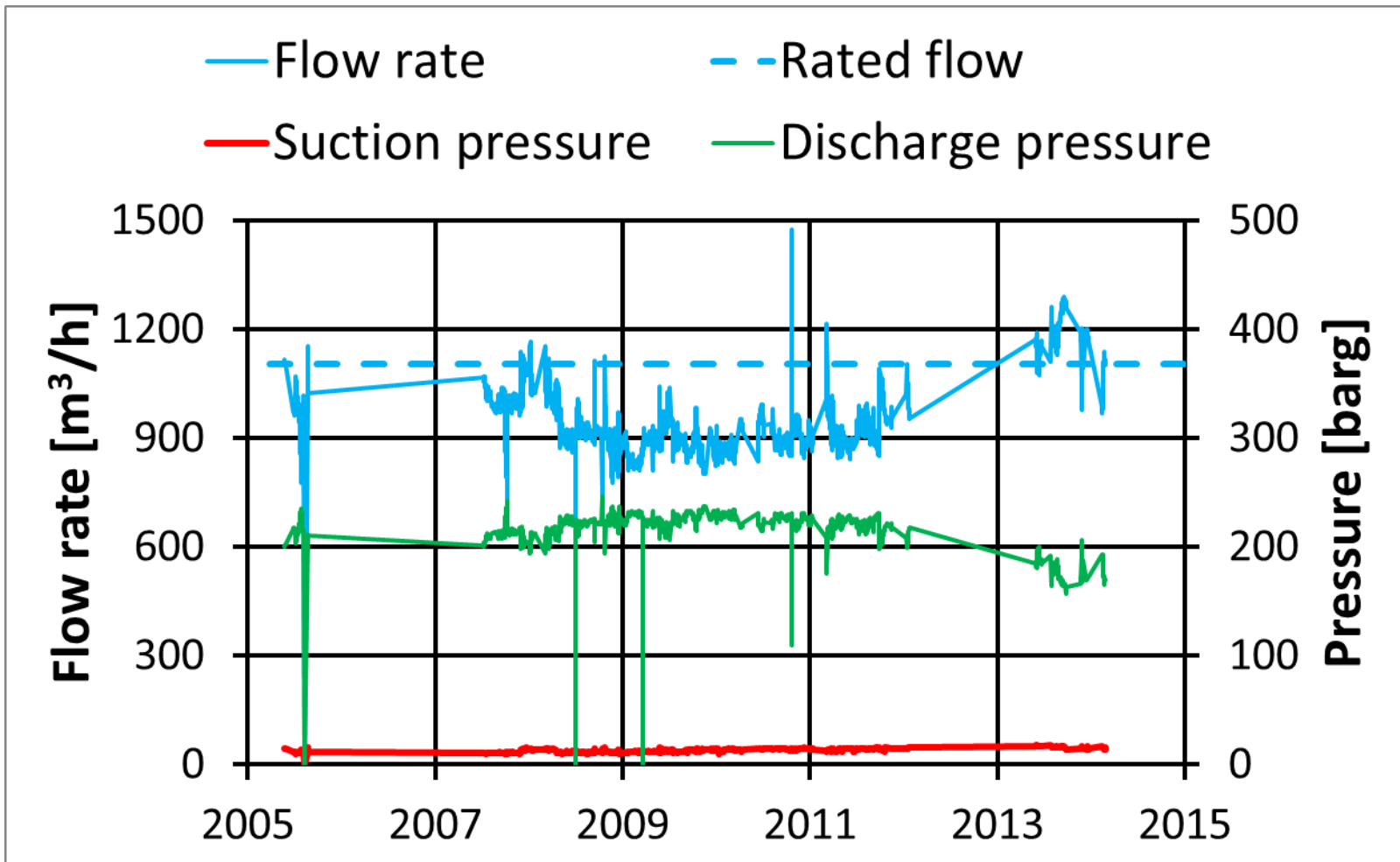


# Introduction – Pump Info/History

	Design	Operating range		
Flow	1104 (4861)	820 (3610)	1000 (4403)	[m <sup>3</sup> /h] ([USGPM])
Head	1966 (6451)	2303 (7556)	2060 (6759)	[m] ([ft])
Temperature	74 (165)	79 (174)	79 (174)	[°C] (°F)
Power	8203 (11000)	6711 (9000)	7084 (9500)	[kW] ([hp])
Speed	4964	4964	4964	[RPM]
U <sub>EYE</sub>	54.5 (179)	54.5 (179)	54.5 (179)	[m/s] ([ft/s])
NPSHA	98.8 (324)	106.7 (350)	121.9 (400)	[m] ([ft])
NPSH3	33.5 (110)	15.2 (50)	22.7 (75)	[m] ([ft])
NPSHA/NPSH3	2.95	7.0	5.33	[-]
NPSH <sub>40000hrs</sub>	45.7 (150)	-	-	[m] ([ft])



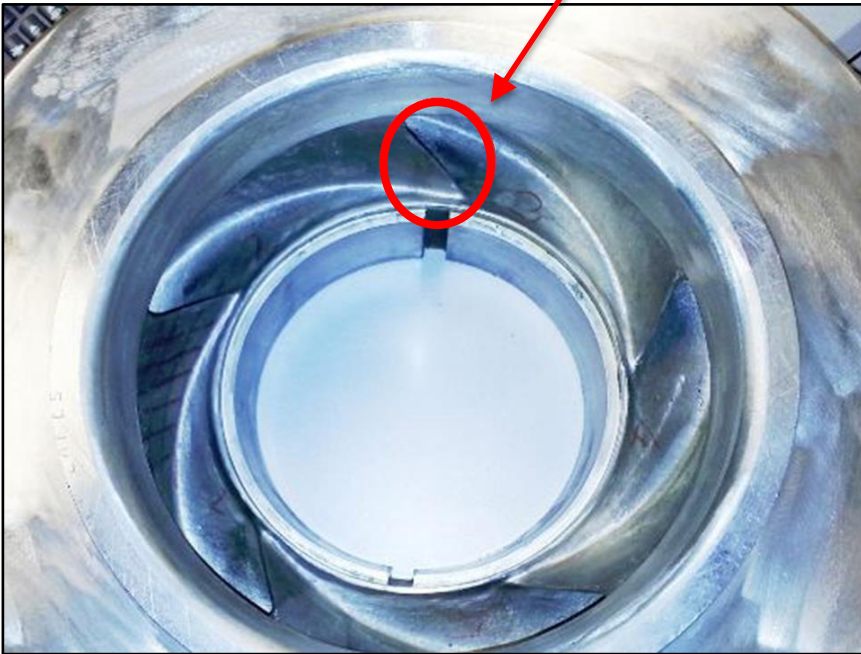
# Introduction – Field Operating Data



# Introduction – Field Experience

**2003 (FAT):**

Vane leading edge reworked to meet rated NPSH3.



**2008 (28000 hrs of operation):**

➤ First stage impeller shows **cavitation** damage.





# Introduction – Field Experience

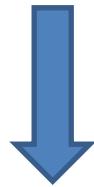
**2008 (First fix):**

Upgrade impeller material from **316 stainless steel** to **Nitronic 50**

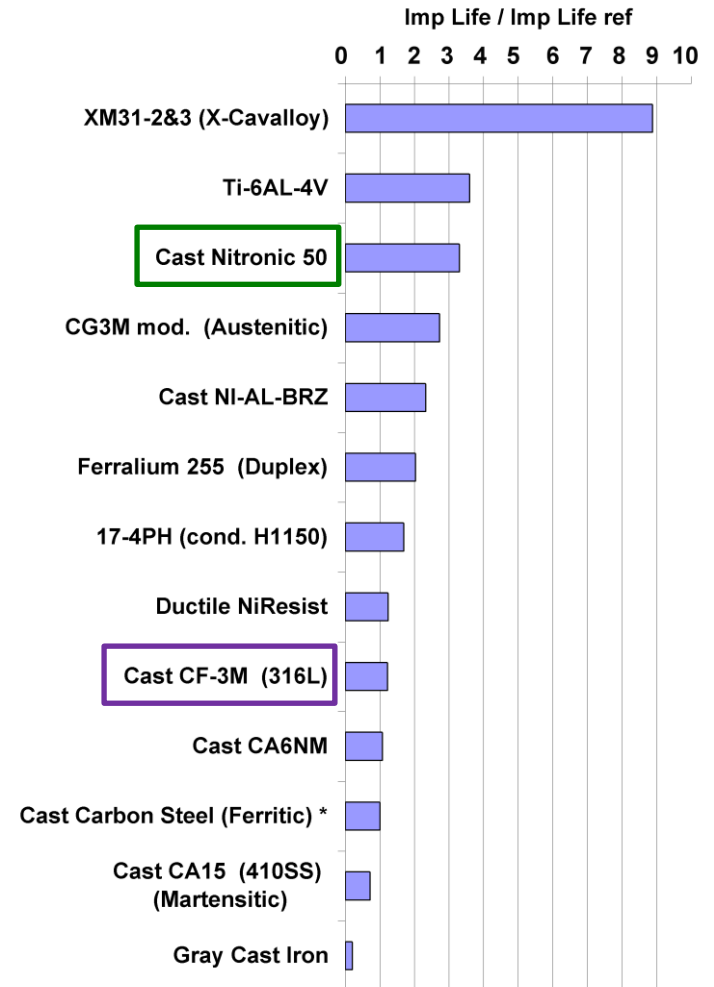
**2012 (13000 hrs operation):**

Nitronic 50 impeller shows same damage pattern

➔ material upgrade did not work



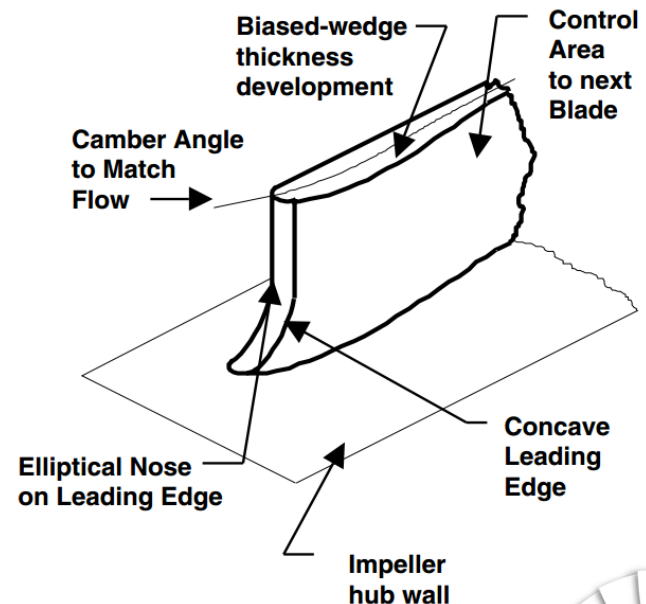
**First-stage impeller needs to be redesigned to remedy the problem**



# Impeller Redesign

- **Key indicator** to drive the redesign process was the **incipient cavitation** Net Positive Suction Head characteristic (NPSHi)
  - NPSHi = level of NPSHA where the first vapor cavities appear (somewhere inside the pump/impeller)

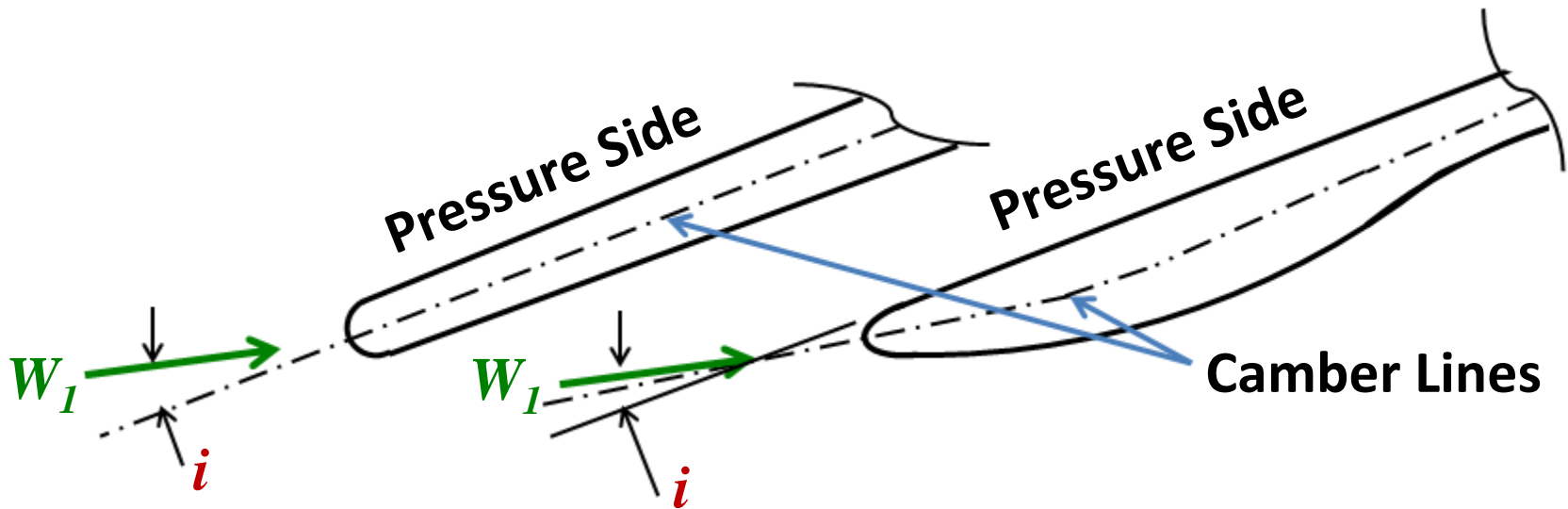
- **Key feature** to implement was a so-called **biased-wedge** blade thickness distribution, shaping the blade leading edge like an airfoil in order to have **minimum pressure drop** at the nose



# Impeller Redesign

Conventional blade  
(symmetrical leading edge)

Biased-wedge leading edge



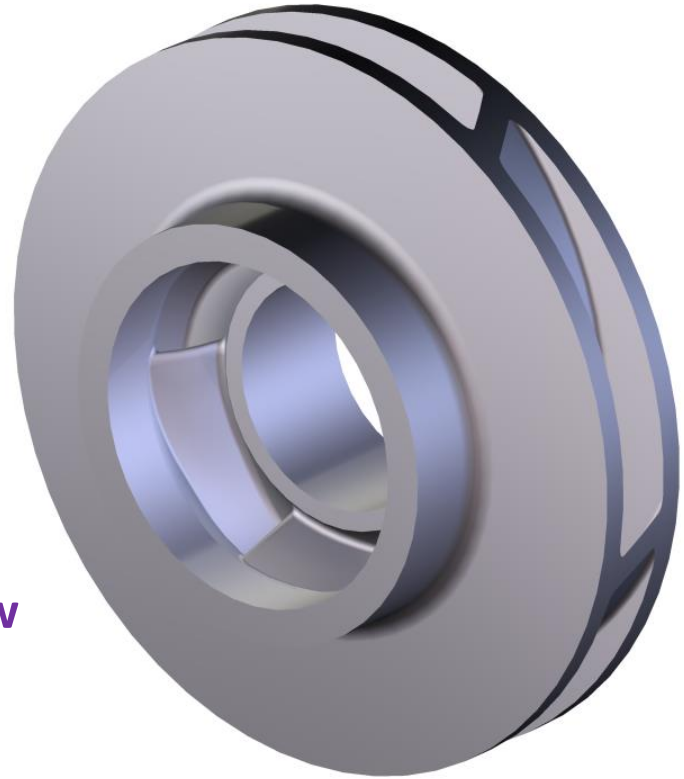
Improved **incidence** between  $W_I$  and the camber line

# Impeller Redesign

## New design:

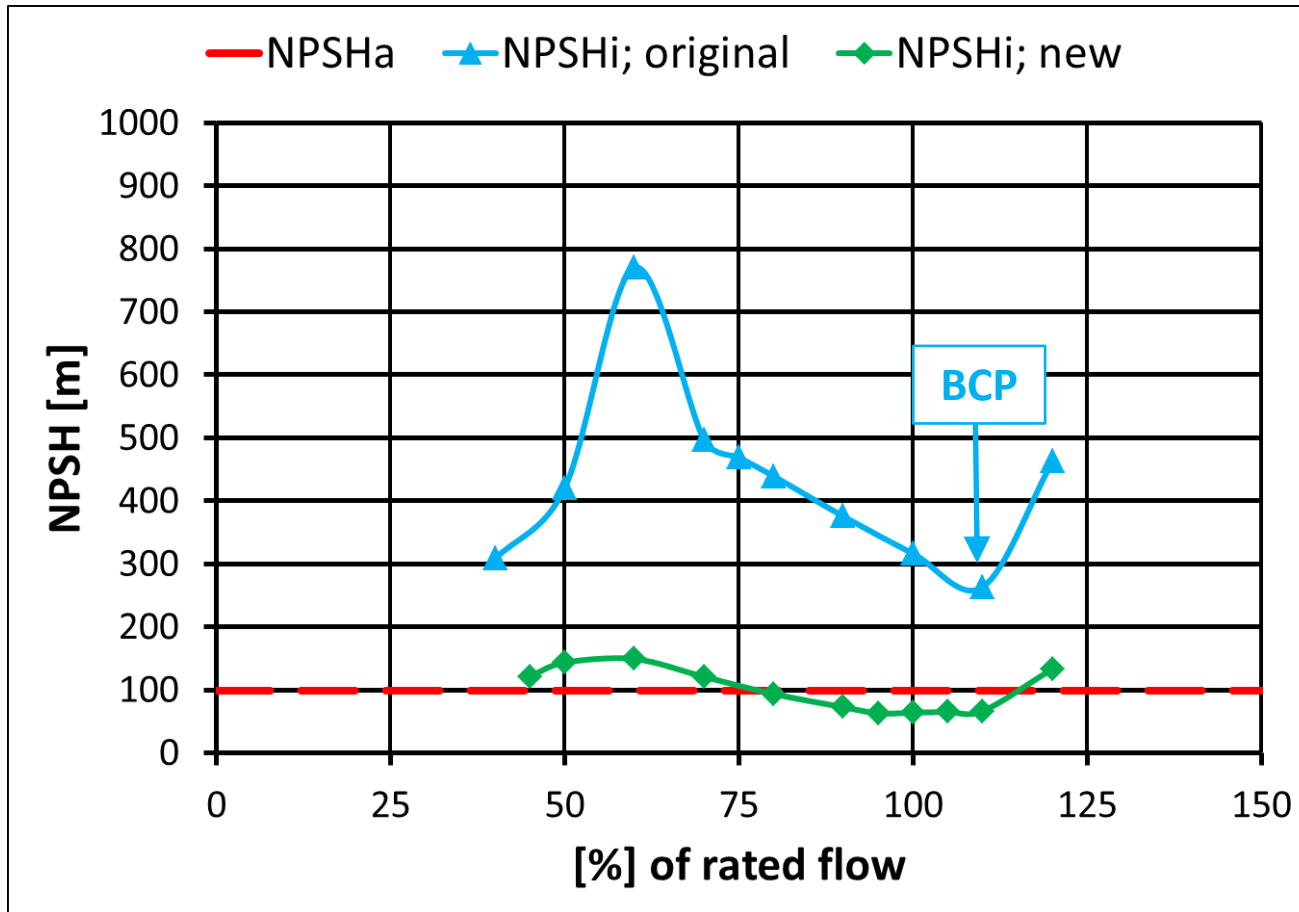
- Based on same parent impeller hydraulic as original
- **Five design iterations** were performed using in-house design tools driven by (CFD) NPSHi calculations
- Design decision criteria:
  - **BCP\* near rated flow** ( $BCP > Q_{rated}$ )
  - **NPSH3 < NPSHA @ max operating flow**
- Rapid prototyping (3D printing) was used to produce the final design

\*BCP = Best Cavitating Point

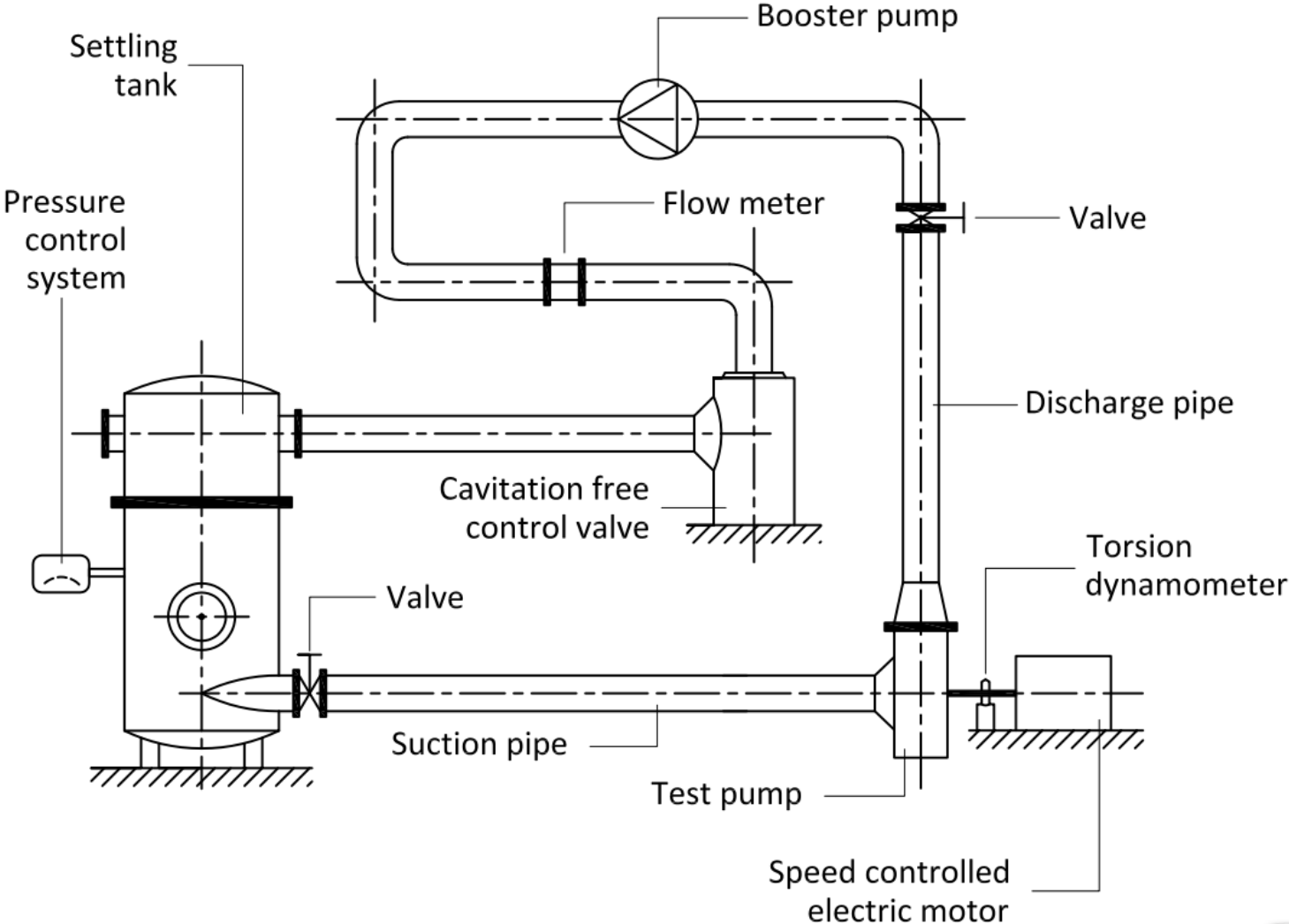


# Impeller Redesign

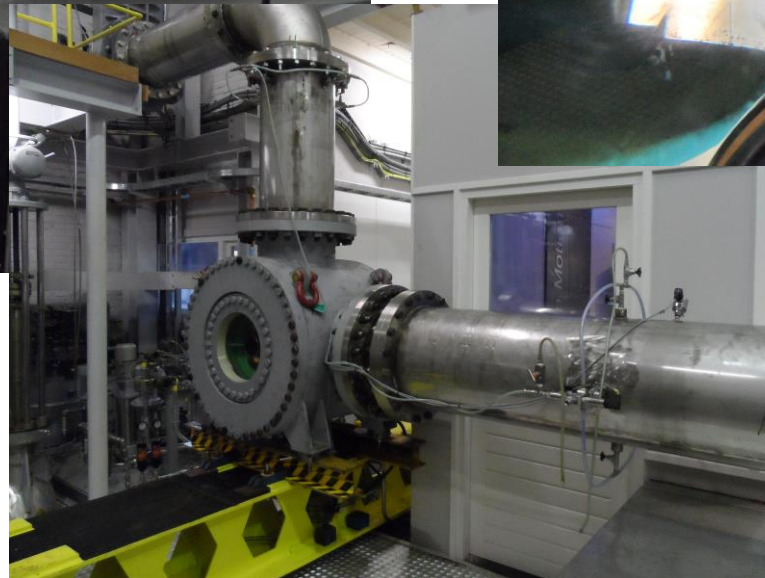
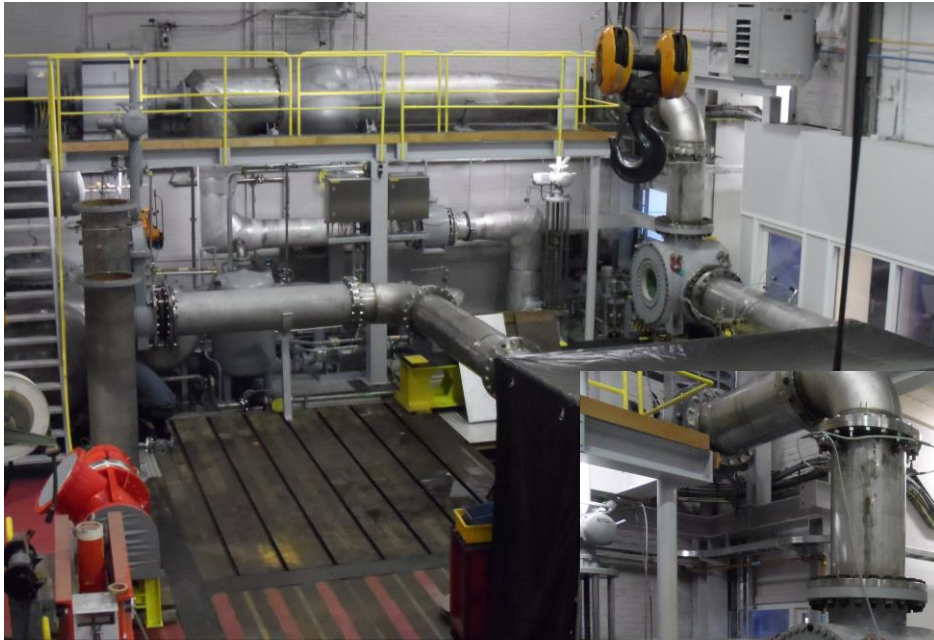
## Results:



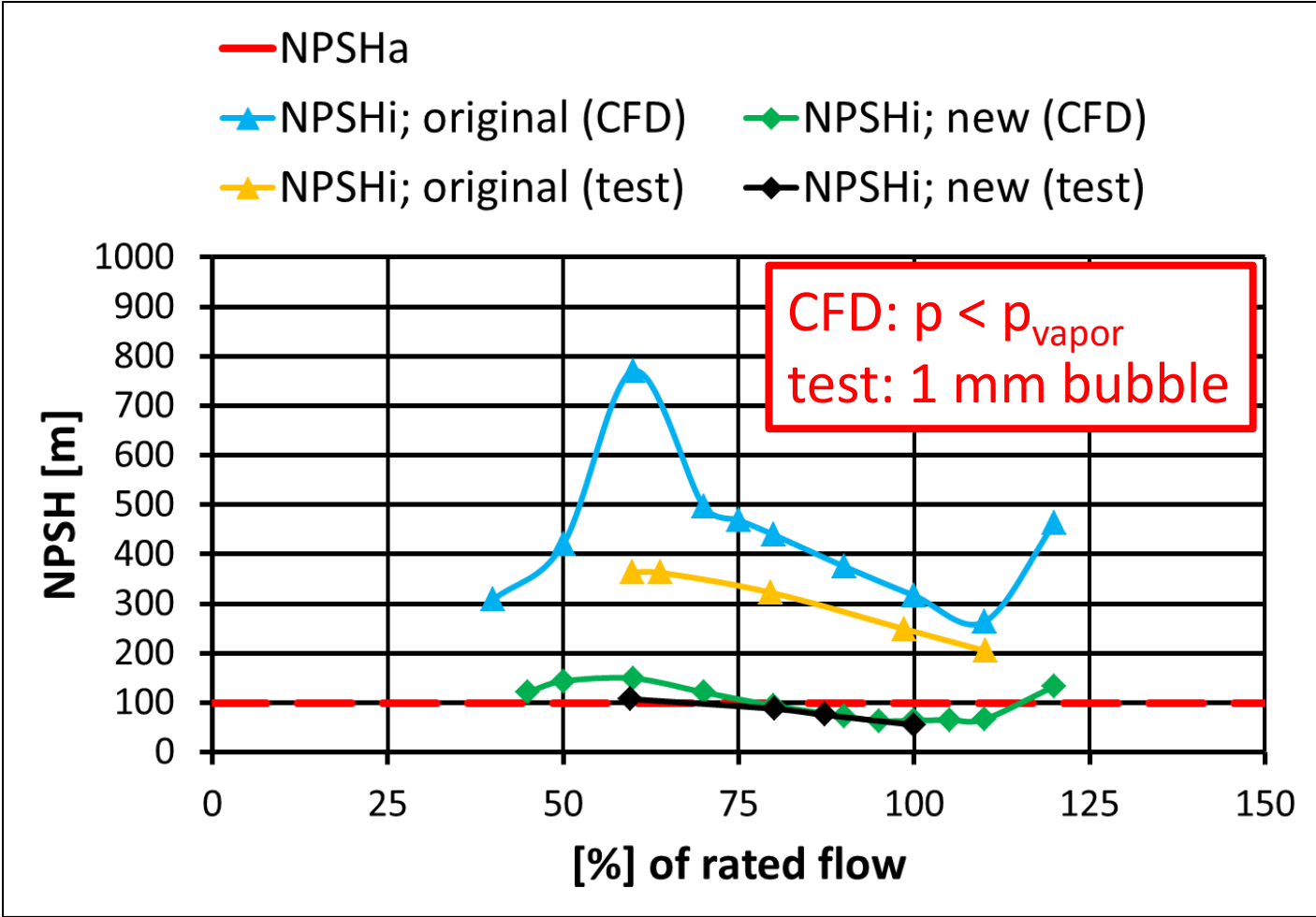
# Verification – Flow Visualization test



# Verification – Flow Visualization test



# Verification – Flow Visualization test





# Verification – Flow Visualization test

Original



New



Results @ rated flow and field NPSHA



# Conclusion - Project results

- Impeller blade leading edge has been optimized to reduce (eliminate) cavitation.
- The improvement has been verified by flow visualization testing.
- Incorporating the **biased-wedge** design has resulted in:
  - **No cavitation from 70% to 100%+ of rated flow @ field NPSHA**
  - Only marginal cavitation ( $L_{cav} < 5$  mm) will be present between 60% - 70% of rated flow
  - Impeller life criterion of 40.000 hours will be (easily) met, and exceed by far
- Pump with new first stage impeller was commissioned successfully in Q1 2015












# Conclusions – Lessons learned

- A biased-wedge design can improve cavitation characteristics significantly, and even eliminate the occurrence of cavitation, for a wide range of flow rates
- The classical 3% head drop criterion is **inappropriate** for high energy pumps in water applications when having high impeller-eye peripheral speed
- Despite operating well above NPSH<sub>3</sub>, with a rated NPSH<sub>3</sub> margin of nearly 3 (!) cavitation damage can occur, limiting impeller life below acceptable.



# Conclusions – Lessons learned

➤ Revised customer standard:

Pump type	NPSH3 curve	40000 hrs curve	Visualization test	Acceptance criteria
All pumps				NPSH Margin MCSF* - 120% > 1 [m] (3.3 [ft])
High energy water injection pumps				Depth of penetration < 75% of vane thickness
Water injection pumps eye peripheral velocity > 47 [m/s] (154 [ft/s])				Depth of penetration < 75% of vane thickness



Required



Not required

\*Minimum Continuous Stable Flow



# Thank you for your attention.

## Questions?

### For further reading:

Van den Berg, N., Bastiaansen, T., and Elebiary, K. 2015 “Predicting, improving and visualizing cavitation characteristics of first-stage impellers in high-speed, high-energy pumps.” ASME-JSME-KSME Joint Fluids Engineering Conference, July 26-31, Seoul, Korea, Paper No. AJK2015-33420

