

Effect of Geometrical Parameters of Aluminum-to-Steel Bolted Connections

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31 **ABSTRACT**

32 Through experimental and numerical studies, this research work aims to provide directions
33 on the optimal geometric configuration for single-lap and double-lap bolted connection
34 between aluminum alloy 6061-T6 and steel. From experimental test results, the effects of
35 different geometric parameters on the joint strength were discussed. These parameters include
36 the end-distance, the side-distance, the pitch-distance, the plate thickness and the joint
37 eccentricity. Then, the experimental results were compared to predicted results using design
38 references and geometric recommendations proposed by design references were critically
39 examined. The experimental study was complemented by finite element (FE) analysis to
40 extend the study to a larger range of parameters. In addition to the analysis of the geometric
41 parameters listed above, the effects of the gage-distance on the joint strength were studied in
42 the FE analysis. The experimental and finite element results show that a careful selection of
43 geometric parameters can result in the high improvement of the connection strength and
44 failure mode. Limiting the side-distance to the minimum recommended value was found to
45 limit the strength of a connection with two bolts in a column to that of the one-bolt
46 connection. In most cases, bearing was found to govern the strength of the connections. The
47 calculated bearing strengths were found to underestimate significantly the connection
48 strength. Based on these analyses, maximum geometric parameters beyond which there is no
49 further increase of the joint capacity were evaluated and optimum geometric parameters were
50 proposed.

51 **Keywords:** Bolted connection, aluminum, geometric parameters, single-lap, double-lap,
52 design prediction, finite element.

53 **1. Introduction**

54 This study was initiated in the context of developing a high strength and low weight portable
55 emergency bridge for railways. The use of aluminum members is a promising option for

56 secondary elements in such bridges due to their light weight, corrosion resistance, and low
57 maintenance cost. Aluminum alloy 6061 is easily available and used for many applications. It
58 can be employed without painting if exposed to general atmospheric corrosion. However,
59 when aluminum is attached to steel components, painting the steel components and placing
60 an electric isolator in the joint are required to prevent galvanic corrosion. A critical aspect to
61 consider for the efficient use of aluminum for portable structures is the behavior of
62 aluminum-to-steel connections. In particular, designers would need to optimize the geometric
63 configurations of those connections.

64 1.1. Research objective

65 The main objective of this research is to provide basic information on the static behavior of
66 aluminum to steel joint in civil engineering applications, compare the experimental results to
67 predicted results using design references, critically examined the geometric recommendations
68 proposed by design reference and to evaluate the maximum geometric parameters beyond
69 which there is no further increase of the joint capacity. This paper is divided into four parts:
70 In the first part, a literature review of aluminum bolted joint is presented. Then, the static test
71 of single-lap and double-lap bolted joints using varying geometric parameters in one and two
72 bolts joints is experimentally studied. Next, calculated joint strengths using equations from
73 three different references, namely Eurocode 9 (EC9) [1], AASHTO LRFD Bridge Design
74 Specifications (AASHTO) [2], and Strength design in aluminum/Commentary on CSA S157-
75 05 (CSA-S157) [3] are compared to experimental results. Finally, finite element analysis
76 validated by experimental results is used to extend the data to a large range of parameters.
77 Based on these different analyses, optimum geometric parameters are proposed for one-bolt
78 and two-bolt connections.

79 **2. Literature review**

80 Among the different joining techniques, bolting appears to be the most practical choice for
81 emergency portable bridges as it can be easily disassembled. However, bolts introduce stress
82 concentration around the bolt-hole where failure can be initiated. When the components are
83 axially loaded, bearing failure, shear tear-out failure, rupture of the net-section, rupture by
84 block shear, yielding of the gross-section and shear failure of the bolt are the six possible
85 modes in which the loaded members can fail. Figure 1 illustrates these six possible failure
86 modes. Equations to calculate the strength for all these failure modes are given in design
87 references. The three design references of interest in this study are: EC9 [1], AASHTO [2]
88 and CSA-S157 [3].

89 Figure 2(a) presents an example of bolted connections loaded in single-lap and double-lap
90 configurations. In a bolted connection, geometric parameters include: the number of shear
91 planes x , end-distance e , side-distance s , width w , pitch-distance p , gage-distance g , plate
92 thickness t , bolt-hole diameter d_h , bolt diameter d , the number of bolts in the row n , the
93 number of bolts in the column m , and the total number of bolts in the connection N . Design
94 references recommend minimum and maximum values for these parameters for the
95 connection of aluminum sections. Table 1 summarises these values for e , s , p and g . The
96 recommended minimum distances may be governed by the clearance for bolt heads and
97 driving tools, as indicated in CSA-S157 [3] while, the maximum are governed by local plate
98 buckling and plate thickness. As presented in Table 1, these maximum and minimum can
99 differ from one design reference to another.

100 The main objective of the experimental studies available in the literature is the prediction of
101 the behavior of the aluminum connection for different configurations. Menzemer et al. [4]
102 performed an experimental investigation to establish criteria to estimate the block shear limit
103 state of bolted connection elements. Block shear failure was found to occur by a tensile
104 fracture in the gage-distance of the first inner row and a more gradual shear failure along each

105 bolt column. Based on their experimental results, an approach to incorporate the connection
106 length into the effective shear strength was proposed. In another paper, Menzemer et al. [5]
107 investigated the bearing strength of three aluminum alloys (5052-H32, 5454-H34 and 3003-
108 H16). Among the key finding, the average ratio of bearing strength to material tensile
109 strength (σ_b/f_u) was higher in configurations with inner aluminum plate than configurations
110 with aluminum outer plates. The effect of bolt pre-load was more relevant in configurations
111 with inner aluminum plate than in configurations with two outer aluminum plates. However,
112 the clamping effect provided either by the bolt pre-load or by the rigidity of the steel cover
113 severely limited the permanent hole elongation. Bearing performance of aluminum bolted
114 connections was also investigated by Wang et al. [6], this time for alloys 6061-T6 and 6063-
115 T5. Bolt-diameter and end-distance were the two parameters studied in the experimental
116 analysis while numerical simulations included also the study of plate thickness. Bearing
117 strength of aluminum was found to be directly proportional to bolt-diameter and sheet
118 thickness. With the increase of end-distance, failure mode was found to change from shear to
119 bearing failure. Bearing strength was linearly proportional to the end-distance up to $e=3d$.
120 Above this value, bearing strength increasing rate was slower. Based on these results, a
121 simplified relation was proposed to evaluate the impact of the end-distance on bearing
122 strength. The effect of end-distance was also investigated by Kim et al. [7] for single-lap
123 aluminum 6061-T6 plates joined with four bolts all arranged in 2 rows and 2 columns. The
124 joint strength was found to increase with the end-distance up to $5d$. Above this value, a
125 reduction of the joint strength was observed due to the excessive out-of-plane deformation
126 (*curling*). Compared to experimental results, the block shear strengths calculated using AISI-
127 S100 [8] and ADM [9] design specifications were underestimated while for curled specimens
128 the design specifications tended to overestimate the joint strength due to the reduction of the
129 capacity caused by curling. Following their experimental study, Kim et al. [10] also

130 performed a numerical simulation to estimate the structural behavior such as ultimate
131 strength, fracture and out of plane deformation (*curling*) effect on single-lap aluminum bolted
132 connections. From their parametric study, the necessity to consider this effect in the design of
133 joint strength with long end-distance and side-distance was highlighted. From Tinl et al. [11]
134 study, the average bearing ratio (σ_b/f_u) at failure of aluminum alloys 5052-H32 and 6061-T6
135 was found to be equal to 1.6 for $e = 1.5d$ and 1.2 for $e = 1.25d$. Compared to the design
136 provisions given by ADM [9] the experimental strengths were found to be larger than the
137 calculated bearing strength using guaranteed minimum mechanical properties.

138 In summary, many values for the spacing of bolts in connections between aluminum plates
139 have been recommended and can also be found in standards. However, except for the end-
140 distance, maximum geometrical parameters beyond which there is no further increase of the
141 joint capacity are not clearly identified. For instance, Kim et al. [10] study indicates that in a
142 four-bolt connection, there is no gain in capacity beyond $5d$. This suggests that there is still a
143 need to further investigate the geometrical configuration of the joints, especially the
144 maximum distances between bolts, or between bolts and plate edges, in order to optimize the
145 strength of the connection.

146 **3. Experimental program**

147 3.1. Overview

148 The static test of few experimental configurations was performed in order to understand the
149 behavior of aluminum to steel bolted connections and to obtain sufficient data to validate the
150 FE model. For this investigation, connections with one-bolt loaded in single-lap and double-
151 lap configurations and two-bolt in a column loaded in single-lap configurations were tested.
152 Five configurations were tested for one-bolt single-lap joint. For the first three configurations,
153 s and e equal $1.5d$ with plate thickness (t) equal to 3.175 mm (T3), 6.35 mm (T6) and 9.525
154 mm (T9). For the fourth configuration, s and e were taken at $1.5d$ and $2d$ respectively with

155 $t=6.35$ mm while for the fifth configuration s and e equal $2.5d$ and $t=6.35$ mm. Results were
156 used to investigate the effect of s , e and t on one-bolt connections. Then, the effect of joint
157 eccentricity on one-bolt connections was studied by comparing results from the single-lap to
158 those of double-lap with the same end-distance, side-distance and plate thickness ($e=1.5d$,
159 $s=1.5d$, $t=6.35$ mm). Finally, the effect of s on the two-bolt single-lap joint strength was
160 studied. A configuration with $e=3d$, $s=2.5d$ and $p=3d$ was compared with another
161 configuration for which $e=2.5d$, $s=3d$ and $p=3d$ both having $t=6.35$ mm. Specimens were
162 named with respect to their respective geometric parameters. For example, S15E15T9 stands
163 for single-lap one-bolt joint with $s=1.5d$, $e=1.5d$ and 9.525 mm thick plates. DS15E15T6
164 stands for double-lap with $s=1.5d$, $e=1.5d$ and 6.35 mm thick plates. S25E30P30T6 stands for
165 single-lap two-bolt in a column with $s=2.5d$, $e=3d$, $p=3d$ and plate thicknesses of 6.35 mm.
166 S30E30G60T3 stands for single-lap two-bolt in a row with $s=3d$, $e=3d$, $g=6d$ and plate
167 thicknesses of 3.175 mm. For all these configurations, ASTM A325 bolts with 12.7 mm
168 diameter and nominal washer on both sides were used. The bolt-hole diameter d_h was
169 approximately equal to 14 mm. Five specimens per configuration were tested except for
170 S30E30G60T3. Three specimens were tested for this configuration for a total of 43 tests.

171 3.2. Experimental setup and measurements

172 The tests were conducted up to failure of the joint in shear using a 500 kN hydraulic testing
173 machine. Figure 2(b) presents the test set-up of aluminum-to-steel connection. The end
174 connections were designed to make the loading axis to coincide with the interface of the two
175 plates so the bolt was primarily loaded in shear. ASTM A325 bolts with 12.7 mm diameter
176 and nominal washer on both sides were used. The length of the bolt was selected to exclude
177 threads from the shear interface. The bolt was tightened to a snug-fit condition, which
178 referred to the maximum effort of a technician using an ordinary wrench. Therefore, the
179 specimen was considered to be bearing type connection. The load was applied at the rate of 1

180 mm/min and the load and displacement were recorded by the control system of the universal
181 testing machine.

182 3.3. Tensile tests of the materials

183 The bolted specimens were taken from different flat bar lots. Flat bars of 63.5 mm and 76.2
184 mm width had a 6.35 mm thickness; flat bars of 38.1 mm width had three different
185 thicknesses: 3.175 mm, 6.35 mm and 9.525 mm. The flat bar of 152.4 mm width had 3.175
186 mm thickness. Two or three coupons per flat bars were used to investigate their mechanical
187 properties. For aluminum 6061-T6 flat bars, a total of 19 coupons were prepared and tested in
188 tension according to the American Standard Test Method B 557M-02 (ASTM B557-02) [12].
189 For the 350W steel flat bars, 12 coupons were prepared and tested in tension according to the
190 American Standard Test Method A370-12 (ASTM A370-12) [13]. All tested coupons had a
191 reduced section. Strain was measured by an axial extensometer with a gage length of 25 mm
192 located at the middle of the reduced section.

193 **4. Experimental results**

194 4.1. Coupons tensile tests results

195 The mechanical properties of tested aluminum coupons are summarized in Table 2. Stresses
196 were computed from the measured loads data divided by the net-section area of each coupon.
197 The elastic modulus (E) of each coupon was calculated by taking the slope of the elastic
198 portion on its stress-strain curve. Since aluminum does not exhibit a flat plateau at the yield
199 strength as is the case of mild steel, ASTM B 557M-02 [12] suggests to determine the
200 aluminum yield strength ($f_{0.2}$) by the offset method at an offset of 0.2%. As presented in table
201 2, the strain at maximum tensile stress (ϵ_{fu}) is approximately 10% while, the strain at $f_{0.2}$ ($\epsilon_{0.2}$)
202 is between 0.40% and 0.62%. The ultimate tensile strength corresponds to the maximum
203 recorded strength. For coupons Al_{A-1} and Al_{B-2} , it was difficult to evaluate the yield strain
204 and/or yield strength and the elastic modulus because the acquisition of these curves was

205 faulty. Tensile strength and elastic modulus reported by design references are also presented
206 in Table 2. Commentary in section 4.2.1 of CSA-S157 [3] recommends for design purposes,
207 that f_u and $f_{0.2}$ shall be the minimum value specified for alloy in the Aluminum Standards and
208 Data Publication (ASD) [14]. For aluminum alloy 6061-T6, this minimum corresponds to that
209 of “shapes” which is 260 MPa and 240 MPa for f_u and $f_{0.2}$ respectively. This value is
210 considered regardless if the material is a plate, shape or sheet. Such recommendation is not
211 specified in [1] and [2]. Therefore, f_u and $f_{0.2}$ were taken as provided in Table 3.2a of [1] and
212 Table 7.4.2.1.1 of [2] for aluminum alloy 6061-T6 “plates”.

213 For mild steel, yield point is simply the first stress in a material, less than maximum
214 attainable stress, at which an increase in strain occurs without increase in stress [15]. The
215 material’s properties of the 5 different lots were similar. The average ultimate tensile strength
216 and average yield strength were approximately 540 MPa and 370 MPa respectively. ASTM
217 A325 bolts were not tested because the strength of the whole connection was not affected by
218 the capacity of the bolt. Its nominal guaranteed tensile strength is 825 MPa and the nominal
219 shear strength for a bolt of 12.7 mm diameter is approximately 63 kN considering the shear
220 strength equals to 0.6 times the nominal tensile strength [15].

221 4.2. Results of bolted connections

222 The average experimental failure loads, P_{exp} , standard deviation, $Std.$, and failure modes,
223 FM , for each configuration are presented in Table 3. It can be observed that the average
224 maximum experimental loads for each configuration of single-lap one-bolt joint are
225 approximately 40 kN, 53 kN and 69 kN for S15E15T6, S15E20T6 and S25E25T6
226 respectively. These load values correspond to 63%, 84% and 109% of the nominal capacity of
227 the 12.7 mm A325 bolt in shear which is 63 kN. By increasing the plate thickness to 9.525
228 mm (3/8 in) with $e=1.5d$ and $s=1.5d$, the average recorded load for S15E15T9 equal 57 kN
229 which is 91% of the nominal capacity of the bolt. For two-bolt joints in a column, the average

230 experimental failure loads are 95 kN and 120 kN for S25E30P30T6 and S30E30P30T6
231 respectively. These loads values correspond to 75% and 95% of the nominal capacity of the
232 two bolts. When the two bolts are arranged in one-row, the average failure load is
233 approximately 87 kN for S30E30G60T3 which represent 69% of the nominal capacity of the
234 two bolts.

235 **5. Analysis of experimental results**

236 As all tested configurations did not have the same material tensile strength, the maximum
237 experimental load ($P_{exp.}$) of each tested joint needs to be adjusted for comparison with other
238 test results. For this purpose, the test results were factored by the ratio of the nominal tensile
239 strength required by CSA-S157 [3] design reference ($f_{uCSA-S157} = 260$ MPa) to the average
240 tensile strength of the corresponding coupon, $f_{u coupon}$ (Equation 1).

$$241 \text{ Factored Load } (P_f) = P_{exp.} \times \frac{f_{uCSA-S157}}{f_{u coupon}} \quad (1)$$

242 These factored loads were used to investigate the gain in joint strength between different joint
243 configurations. In Figures 3 to 6, the Factored load was used instead of the experimental load
244 because it allowed to better compare the force-displacement curves for the various
245 configurations.

246 5.1. Effect of side-distance and end-distance on a one bolt (1X1) single-lap configuration

247 Figure 3(a) presents the typical failure mode of single-lap aluminum-to-steel one-bolt joints.

248 In this figure, the aluminum plate is on the bottom side while the corresponding steel plate is
249 on the top. The five specimens of each configuration experienced the same failure mode. For

250 S15E15T6, shear tear-out failure of the aluminum plate was observed (Figure 3a). With the

251 increase of the end-distance (S15E20T6), the aluminum plate experienced rupture of the net-
252 section (Figure 3b). A further increase of the end-distance associated with an increase side-

253 distance (S25E25T6) changed the failure mode of the joint to a bearing failure of plate

254 around the bolt-hole in the loading direction (Figure 3c). This elongation, although similar to

255 that of S15E15T6 configuration, was assumed to be bearing failure. This assumption was
256 based on the fact that CSA-S157 [3] recommends that “for force directed towards an edge
257 distance, the capacity is governed by shear failure in the connected material up to an end edge
258 distance in excess of two diameters”. With S25E25T6 configuration, a limited yielding of the
259 bolt was also observed. Furthermore, the steel plate reaches its plastic state and an ovalization
260 of the bolt-hole in the steel plate was observed.

261 In Figure 3(d), factored typical force-displacement curves of the three configurations are
262 compared. It is observed that the factored load (P_f) increased by 32% and 61% for S15E20T6
263 and S25E25T6 respectively as compared to S15E15T6. For design consideration, design
264 references such as [1] and [3] suggest to limit the bearing strength of a connection with $e > 2d$
265 to that of approximately $2d$. Experimental results shows that this restriction is acceptable for s
266 lower than $2d$. Above this, such restriction underestimates the bearing strength of a one-bolt
267 connection. An observation of the displacement at which failure occurred in the S15E15T6
268 and S15E20T6 shows that increasing the joint end-distance only did not increase the joint
269 displacement at failure. However, the displacement increased to approximately 10 mm when
270 both the end and side distances were increased to $2.5d$. As S25E25T6 leads to bearing failure
271 and to a higher joint strength, it was taken as a reference baseline for two-bolt connections.

272 5.2. Effect of plate thickness in a one-bolt single-lap configuration

273 The effect of plate thickness was investigated by comparing one-bolt connections with same
274 end and side distances but different plate thicknesses. Figures 4(a), 4(b) and 4(c) present the
275 typical failure mode of specimens S15E15T3, S15E15T6 and S15E15T9 respectively. Results
276 indicate that plate thickness has no effect on the joint failure mode. For this configuration,
277 failure by shear is the observed mode for the three connections.

278 Figure 4(d) presents the typical force-displacement curves of the three tested configurations.
279 The average factored loads of S15E15T3, S15E15T6 and S15E15T9 are 18.8, 33.8 and 47.5

280 kN respectively. The relationship between the factor load and the plate thickness appears to
281 be not perfectly linear. This effect will be studied further in section 8.1. With S15E15T3, the
282 displacement at failure is approximately 3 mm while with S15E15T6 and S15E15T9 the
283 displacement at failure is around 7 mm.

284 In Table 3, it can be noted that the factored load obtained with S15E15T9 is only 1.06 times
285 higher than that obtained with S15E20T6 which is 1.5 times thinner but with $e = 2d$. This
286 suggests that a careful selection of the geometric parameters can result in large gains in
287 material weight and cost.

288 5.3. Effect of joint eccentricity

289 Joint eccentricity is usually present in single-lap configuration resulting in out-of-plane
290 deformation. Plate end-curling observed as a result of such deformation is more pronounced
291 in thinner plates with longer end-distance than shorter ones as illustrated in Figure 5(a) and
292 5(b). It can be prevented by using an even number of shear planes. The effect of joint
293 eccentricity in the joint axial strength was evaluated by comparing the single-lap S15E15T6
294 with the double-lap (steel-aluminum-steel) DS15E15T6. In Figures 5(c) and 5(d), failure
295 mode of S15E15T6 is compared to DS15E15T6. As it can be observed, restraining the out-of-
296 plane deformation did not change the failure mode of the joint. Shear tear-out damage is
297 observed on the aluminum plate of both DS15E15T6 and S15E15T6. No significant
298 deformation is observed on the steel plates and on the bolt.

299 Figure 5(e) presents the force-displacement curves of DS15E15T6 and S15E15T6. A gain of
300 6% the factored load (P_f) is measured when the eccentricity is restrained. Failure of the
301 DS15E15T6 compared to S15E15T6 occurs earlier in the loading history of the joint. The
302 measured displacement is approximately 5 mm for DS15E15T6 compared to approximately 7
303 mm for S15E15T6. This is probably due to the two outer steel plates. As their rigidity is 3
304 times greater than that of aluminum and due to the restriction of out-of-plane deformation,

305 the ability of the aluminum inner plate to freely deform was reduced. As the strength obtained
306 with the double-lap configuration was only few percent higher than that of single-lap, it was
307 not found necessary to pursue the tests with the double-lap configuration. Therefore, the
308 analysis of the two bolts joint was performed only in single-lap configuration.

309 5.4. Effect of geometric parameters in two-bolt single-lap configuration

310 Figures 6(a), 6(b) and 6(c) present the typical failure modes of S25E30P30T6, S30E30P30T6
311 and S30E30G60T3 respectively. For specimens S25E30P30T6 rupture of the net-section was
312 observed at the aluminum first inner row (Figure 6a). The bolt of the inner row shows sign of
313 shear deformation while the corresponding row on the steel plate shows a light bearing
314 damage. Rupture of the net-section occurs also for S30E30P30T6 on the aluminum first inner
315 row with signs of bearing damage on the two holes (Figure 6b). Signs of bearing damage
316 were also present on the two holes of steel plate while the two bolts reached a plastic state.
317 For S30E30G60T3, failure presented in Figure 6(c) occurs by bearing with hole elongation
318 approximatively equal to 2 times the bolt diameter followed by curling of the aluminum
319 plate. In Figure 6(c), out-of plane deformation of the aluminum plate is observed between the
320 two bolts due to the gage distance, $g=6d$, taken above the maximum recommended values of
321 EC9 [1] ($g=3.5d$) and AASHTO [2] ($g=4.25d$) for a 3.175 mm thick plate. Bearing of the
322 steel plate was also observed while; the two bolts showed signs of damage.

323 Figure 6(d) presents the typical force-displacement curves of the three configurations.
324 Maximum displacements of 8 mm and 12 mm are observed for S25E30P30T6 and
325 S30E30P30T6 respectively. A gain of 31% of the factored load (P_f) is achieved when the
326 side-distance is increased to $3d$. The factored joint capacity in one-bolt connection
327 (S25E25T6) is equal to 54 kN. Doubling the number of bolts with the same side-distance
328 (S25E30P30T6) only increased the joint strength by 46% because the capacity was limited by
329 the rupture of the net-section and the end-distance has no effect on the joint strength. A

330 further increase of the side-distance to $3d$ (S30E30P30T6) leads to 91% increase in the
331 factored joint strength. As no fracture was observed between the two bolts in both
332 configurations, the pitch-distance $p=3d$ recommended by [3] is found sufficient to prevent the
333 joint failure on the shear path. Therefore, for the two-bolt in a column with $t=6.35$ mm, a
334 side-distance s greater than $3d$ (or $w > 6d$) is required for the joint to achieve the nominal
335 capacity of the two bolts A325 in shear. This value of s is expected to change with the plate
336 thickness, the number of bolt in the joint, and the bolt size and properties.

337 Figure 7(d) also shows the force-displacement curve for specimen S30E30G60T3. The results
338 of this specimen will be used later to validate the FE model for the two-bolt in a row
339 configuration.

340 **6. Comparison between experimental and predicted results**

341 The test results of one-bolt and two-bolt connections are compared with the nominal strength
342 prediction obtained using equations recommended by [1], [2] and [3]. As only the nominal
343 strength is considered, strength reduction factors are not used for the calculation of the
344 strength predictions.

345 6.1. Design Equations

346 When failure is predicted to occur in the net-section, all three design references establish the
347 corresponding net-section strength (T_r) presented in Equation 2.

$$348 \text{Nominal net – section strength } (T_r) = (w - nd_h)tf_u \quad (2)$$

349 The three design references do not give a clear guideline of the shear tear-out strength (S_r)
350 prediction as it is the case of steel structures. As shear tear-out is a particular case of block
351 shear failure also called tear-out in [3], it was assumed that block-shear strength equation in
352 these design references can also be applied to shear tear-out strength. Therefore, the nominal
353 shear-tear-out strength per bolt was evaluated using Equations 3 to 5.

$$354 \text{CSA – S157 [3] nominal Shear tear – out strength } (S_r) = etf_u \quad (3)$$

355 EC9 [1] nominal Shear tear – out strength (S_r) = $0.58(2e - 0.5d)tf_u$ (4)

356 AASHTO [2] nominal shear tear – out strength (S_r) = $1.16etf_u$ (5)

357 The nominal bearing strength (B_r) per bolt is evaluated using Equation (6) to (8) for [3], [1]
 358 and [2] respectively. It can be seen in Equation (8) that the bearing strength of the material is
 359 required to evaluate the connection bearing strength. This strength was not evaluated for the
 360 coupons tested in this study. Therefore, the ratios of f_{bu}/f_u and f_{by}/f_y calculated from values
 361 provided in [2] design reference Table 7.4.2.1 were used. From the values provided in this
 362 reference, the bearing strength of aluminum alloy 6061-T6 (f_{bu}) equals $2.1f_u$ while the bearing
 363 yield strength (f_{by}) equals $1.66f_y$. These ratios were used in conjunction with the coupon
 364 tensile strength as presented in Equation 8. For $e < 2d$, AASHTO [2] recommends to multiply
 365 B_r by the ratio of $e/2d$

366 CSA – S157 [3] nominal bearing strength (B_r) = $\min \left\{ \begin{array}{l} etf_u \\ 2dtf_u \end{array} \right.$ (6)

367 EC9 [1] nominal bearing strength (B_r) = $2.5\alpha_b df_u$ with $\alpha_b = \min \left\{ \begin{array}{l} \frac{e}{3d_h} \\ 1 \\ \frac{f_{ub}}{f_u} \end{array} \right.$ (7)

368 AASHTO [2] nominal bearing strength (B_r) = $\min \left\{ \begin{array}{l} dtf_{by} = 1.66dtf_y \\ dtf_{bu}/1.2 = 1.75dtf_u \end{array} \right.$ (8)

369 6.2. Analysis of the predicted results

370 Table 3 column 5 to 13 lists the predicted failure modes and failure loads of the three design
 371 references. The material tensile strengths obtained from the tested coupons were used for this
 372 calculation. Therefore, the predicted load P_n should be compared to the predicted
 373 experimental load P_{exp} . The experimental to predicted ratios are also reported in Table 3.

374 For connections S15E20T6, S25E30P30T6 and S30E30P30T6, experimental study reveals a
 375 failure mode by net-section. However, with EC9 [1] and AASHTO [2] bearing strength was
 376 found to govern the design prediction of these three configurations. Failure prediction by
 377 CSA-S157 [3] was consistent with the experimental failure for S15E20T6 and S25E30P30T6.

378 However bearing strength governs the design prediction for S30E30P30T6. With [3], the net-
379 section capacity was also well predicted with a maximum difference of 14% compared to
380 experimental failure load. The strength of these connections was underestimated by 20% to
381 64% compared to experimental results when bearing failure was the predicted mode.

382 For connections S15E15T3, S15E15T6, S15E15T9 and DS15E20T6, it was assumed in the
383 experimental analysis that the observed failure mode is shear tear-out (instead of bearing)
384 because of $e < 2d$ as suggested in [3]. However, with [1] and [2] design predictions, bearing
385 strength was found to govern the design prediction. Failure prediction by [3] was consistent
386 with the experimental failure as the shear strength equation (Equation 3) is equal to the lower
387 bound of bearing strength (Equation 6). With [1] and [2], the predicted strength was found to
388 be very conservative with a maximum difference of 60% compared to experimental results.
389 As for [3], a better prediction of the strength was obtained with a maximum difference of
390 20% compared to experimental results.

391 For connections S25E25T6 and S30E30G60T3, all three design references predicted a
392 bearing failure which is consistent with the experimental failure mode. However, the bearing
393 strength prediction of these three references underestimated the connection strength by 30%
394 to 106% as compared to experimental results. This is due to the fact that design references
395 limit the bearing strength to approximately twice the ultimate tensile strength. For $e > 2d$, such
396 limitation underestimates the connection strength.

397 More data are required to better comprehend the relationship between different geometric
398 parameters and the joint strength and to develop design equations capable of providing an
399 accurate prediction of the joint strengths. Finite element analysis is a good tool to extend such
400 data.

401 **7. Finite element analysis**

402 7.1. Overview of the finite element model

403 A finite element (FE) analysis was performed to investigate the strength of aluminum bolted
404 joints with the commercial software ADINA 8.7.3. The analysis started with a validation
405 study. The experimental results were used to validate the FE results. This analysis was
406 followed by a parametric study where a 3D FE analysis was used to study the effect of plate
407 thickness and verify the assumption made in section “Effect of plate thickness”. For this
408 verification, $t=3.175$ mm, 6.35 mm, 9.525 mm, 12.7 mm, 15.875 mm, 19.05 mm, 25.4 mm
409 with the constant values of $e=1.5d$ and $s=1.5d$ were considered. Then, the effects of
410 increasing the end-distance ($e=1.5d, 2d, 3d, 4d, 5d$) and the side-distance ($s=1.5d, 2d, 2.5d,$
411 $3d, 4d, 5d$) in one-bolt connection were analyzed using 2D FE analysis. Finally, the optimum
412 geometric parameters for two-bolt connections either aligned parallel or transversal to the
413 loading direction were investigated. The end-distance and the side-distance were varied from
414 $1.5d$ to $5d$ as in one-bolt configuration while increasing the pitch-distance ($p=2d, 3d, 4d, 5d$)
415 for two-bolt parallel or the gage-distance ($g=2d, 3d, 4d, 5d$) for two-bolt transversal to the
416 loading direction were considered. In line with the experimental investigation, maximum
417 geometric parameters beyond which there is no increase in the joint strength were evaluated.
418 Results obtained from this analysis were also used to evaluate the optimum parameters of one
419 and two bolts connection for different plate thicknesses, bolt sizes and bolt grades. A linear
420 relationship between the joint strength and both the plate thickness and bolt size was
421 assumed. These optimums were chosen based on the values at which the nominal shear
422 strength of a high strength bolt A307, A325 and A490 is reached. The recommendation of
423 AASHTO [2] to limit the net-area (A_n) of the connecting element to the maximum value of
424 85% the gross-area (A_g) of the plate was also taken in consideration. Although such
425 requirement is not specified for aluminum structures, it was considered for choosing the side
426 and gage distances to make sure an economic design is identified.

427 7.2 Analysis assumptions

428 In the experimental study of the aluminium-to-steel connection, failure of the joint was due to
429 the aluminum fracture. Therefore, to reduce the computation time, only the aluminum plate
430 and bolt were modeled in the finite element (FE) analysis. To ensure that the model reliability
431 will not be affected with such simplification, a 3D model, which includes the aluminum plate,
432 the steel plate and the bolt (FE_3DAS) was also computed. The results obtained with the 2D
433 (FE_2DA) and 3D (FE_3DA) simplified models were compared to FE_3DAS and
434 experimental results. As bolts were tightening to a snug-fit condition (bearing type
435 connection), the bolt preload was neglected. The bolt preload was also not considered in the
436 FE analysis by [10] for the same preload condition. Figures 7(a) and 7(b) present the typical
437 FE_3DAS and FE_3DA models used for this analysis.

438 For model validation, all configurations tested in the experimental program were analysed.
439 Material properties of these configurations are shown in Table 2. In the static environment of
440 ADINA, the aluminum and steel plates were modeled as a material with bilinear elasto-
441 plastic behavior. The 3D solid element with 8 nodes per element and 3 degrees of freedom
442 was considered. The analysis assumptions were large displacement and strain. Full Newton
443 method was used for the iteration. The bolt was modeled as a cylinder of 12.7 mm diameter.
444 Contacts between the bolt and the plates were modeled by contact elements available in the
445 FE software. The contact interface were generated as a pair of surface elements. On this
446 interface, a bolt was defined as a target surface and the bolt-hole as a contactor surface. This
447 assumption was based on the fact that the strength of the steel bolt is greater than that of
448 aluminum and steel plates. No friction was considered on this interface. For the 3D model
449 containing the steel and aluminum plates (3DAS), fixed boundary condition was applied on
450 the aluminum far end plate edge ($X=Y=Z=0$) and a uniform pressure was applied in the
451 longitudinal z axis of the far end plate edge of the steel plate (Figure 7a). For the 3D
452 simplified model containing only the aluminum plate (3DA), fixed boundary condition was

453 applied on the bolt ($X=Y=Z=0$) and a uniform pressure was applied in the longitudinal z axis
454 of the far end plate edge of the aluminum plate (Figure 7b). The external load was
455 incrementally applied on the structure. Once the deformation of the material at f_u (ϵ_{fu}) was
456 reached, the structure diverged. The load to which the structure diverged was taken as the
457 maximum capacity that the joint can support.

458 For the parametric study, $f_{0.2}=279$ MPa, $f_u=310$ MPa, $\epsilon_{0.2}=0.4\%$ and $\epsilon_{fu}=10\%$ were
459 considered. A 3D FE analysis was first used to investigate the effect of plate thickness. Then,
460 to further reduce the computational process, a 2D FE analysis was considered to evaluate the
461 effects of e , s , p and g . In addition to the FE modeling assumptions described above, six
462 degrees of freedom and 9 nodes per element were used for 2D FE model (2DA). The bolt was
463 modeled as a half cylinder and the contact feature available in the software was used. The
464 parameters: e , s , p and g were consecutively changed while $d=12.7$ mm and the input plate
465 thickness ($t=6.35$ mm) were kept constant.

466 7.3 Model validation

467 Figure 7(c) to 7(e) present the post-processing strain distribution of the FE model. Based on
468 the plastic strain distribution along a given failure path of the model, the joint failure mode
469 was defined. For example, for net-section failure, plastic strains are developed across the
470 centerline of the bolt-hole in the net-section path (Figure 7c). For shear tear-out failure
471 presented in Figure 7(d) plastic strains are developed between the side of the bolt-hole up to
472 the free end edge of the plate while in bearing failure presented in Figure 7(e), plastic strains
473 are limited ahead of the bolt hole in the bearing path and barely reach the free end edge of the
474 plate.

475 In Figure 8, the typical force displacement curves obtained in the FE analysis are compared to
476 that of experimental results. In general, no significant difference in the failure loads are
477 observed between the three different models. In Figure 8(a) and 8(b) it can be observed that

478 the elastic portion of the force-displacement history of FE_3DAS is stiffer than that of
479 FE_3DA and FE_2DA. On the other hand, it can be observed that the curves of FE_3DA and
480 FE_2DA in Figure 8(a) to 8(d) are quite consistent with that of experimental results up to the
481 peak load at which the FE models stop. Therefore, the simplified 2D (FE_2DA) and 3D
482 (FE_3DA) models will be used for the rest of the study.

483 In Table 4, the ultimate loads and failure modes obtained from simplified 2D and 3D FE
484 analyses of one and two bolts connections are compared to the average failure loads obtained
485 in the experimental analysis. As it can be observed, both 2D and 3D results are in good
486 agreement with experimental results, FE failure loads are mostly conservative. The ratios of
487 experimental to predicted results are within 11% difference for 3D and 2D models. The
488 observed FE failure modes were also consistent with the experimental failure mode.
489 Therefore, the 3D and 2D models are enough reliable to be used for the parametric analysis.

490 **8. Parametric simulation and analysis of the results**

491 Following the satisfactory agreement between 2D, 3D FE models and experimental results, a
492 parametric study was carried out. The results obtained from the parametric simulation are
493 presented in Figure 10, Tables 5 and 6.

494 8.1. Effect of plate thickness in one-bolt connection

495 The effect of plate thickness on the joint strength were analysed by keeping the material
496 properties ($f_{0.2}=279$ MPa, $f_u=310$ MPa, $\epsilon_{0.2}=0.4\%$ and $\epsilon_{fu}=10\%$) and $e=s=1.5d$ constant with
497 varying values of t ($t=3.175$ mm, 6.35 mm, 9.525 mm, 12.7 mm, 15.875 mm, 19.05 mm, 25.4
498 mm). Results obtained from this simulation are presented in Figure 9. Figure 9(a) shows the
499 plastic strain distributions of S15E15T25, S15E15T12 and S15E15T3. Failure occurs by
500 shear tear-out regardless of the size of the plate thickness. In Figure 9(b), it can be observed
501 that the relationship between the plate thickness and the joint strength is to a certain extent
502 linear in the model. The experimental results obtained for S15E15T3, S15E15T6 and

503 S15E15T9 are also presented in this figure. It was observed experimentally that the
504 relationship between the plate thickness and the average joint strength was not perfectly
505 linear. However, by tracing all the data obtained experimentally for these configurations in
506 Figure 9(b), the linear relationship between the plate thickness and the joint strength is an
507 acceptable simplification. The possible non-linearity of this relationship would need to be
508 studied with additional experimental tests. In particular, for plates with large thicknesses,
509 which are realistic in the context of bridge construction, no experimental data was found in
510 the literature and therefore the numerical results proposed in this study would need to be
511 further studied with experimental tests to confirm the trends observed.

512 For the evaluation of the optimum parameter, the linear relationship between the bolt
513 diameter and the plate thickness was assumed as demonstrated by the experimental study of
514 Wang *et al.* [6] and section 8.1. Results obtained from the FE analysis and presented in Table
515 5 and 6 were used to extrapolate the strength of connections with different plate thicknesses
516 and bolt diameters. Tables 7 to 9 present the proposed optimum for values of $6.35 \text{ mm} \leq t \leq$
517 25.4 mm and $12.7 \text{ mm} \leq d \leq 25.4 \text{ mm}$. In these tables, cells with no value are those from which
518 the plate thickness could not sustain the capacity of the bolts or which required a side-
519 distance larger than the maximum recommended values of both [1] and [2] and from which
520 the ratio of $A_n/A_g > 0.85$. The following paragraphs described some of the important results
521 obtained in the parametric simulation and the criteria behind the choice of optimum
522 parameters.

523 8.2. Optimum end-distance (e) and side-distance (s) in one-bolt connection

524 Figure 10(a) presents the effects of the end-distance (on the positive x-axis) and the side-
525 distance (on the negative x-axis) on the failure load for one-bolt joint as obtained by the FE
526 analysis for $t=6.35 \text{ mm}$ and $d=12.7 \text{ mm}$. The predicted strengths of the one-bolt joint using
527 the maximum and minimum e and s recommended by EC9 [1] were also evaluated and are

528 presented in this figure. These minimum are equal to $1.2d$ for s and e while the maximum is
529 $4t+40$ mm for both s and e which corresponds to $5.15d$ for $t=6.35$ mm and $d=12.7$ mm. FE
530 results show an increase of the joint capacity with the increase of s and e up to a limit value
531 of $3d$ and $4d$ respectively. Table 5 shows the FE results of one-bolt connection. It can be
532 observed that the strength of the connection is governed by net-section failure at $s < e$, shear
533 tear-out failure for $e \leq s \leq 2d$ and bearing failure for e and $s > 2d$. Comparisons between the
534 strength obtained from FE analysis at the limits of $3d$ and $4d$ for s and e respectively reveals
535 that the measured gain is 33% and 75% higher than the predicted strength at maximum and
536 minimum values recommended by [1]. These values are consistent with the maximum values
537 recommended by [1], which are $5.15d$ ($4t+40$ mm) for both s and e . For design consideration,
538 design references such as [3] suggest to limit the bearing strength of a connection with $e > 2d$
539 to that of $2d$. Figure 10(a) shows that this restriction is acceptable for s lower than $2d$. Above
540 this, such restriction underestimates the bearing strength of a one-bolt connection. From
541 Table 5, the values of $s=2d$ and $e=3d$ were found sufficient to improve the connection
542 strength to that of one bolt A325 nominal shear strength with $d=12.7$ mm and $t=6.35$ mm.
543 With the value of $s = 2d$, the ratio of $A_n/A_g = 0.75$ which is lower than the limit ratio of 0.85 is
544 obtained. The optimum parameters of one-bolt with varying bolt sizes, plate thickness and
545 bolt grade are presented in Table 7. It can be observed that optimum values are related to the
546 bolt diameters, bolt grade and plate thickness. For bigger bolt size ($d=25.4$ mm) or A490 bolt
547 grade, these optimums can reach the limit of $3d$ and $4d$ for s and e respectively with A_n/A_g
548 < 0.85 .

549 8.3. Evaluation of the optimum geometric parameters for two-bolt joint in a column

550 Figure 10(b) presents the effects of geometric parameters for the joint with two bolts parallel
551 to the loading direction, as obtained from the FE analysis for $t=6.35$ mm and $d=12.7$ mm. The
552 predicted strengths of the two-bolt joint using the maximum and minimum e , s and g

553 recommended by [1] are also presented. The maximum and minimum e and s are the same as
554 in one-bolt joint while the limit values of p are: $2.4d \leq p \leq 7d$ ($14t$). On the positive x-axis of
555 Figure 10(b), the effect of e with $s=1.5d$, $3d$ and $4d$ and varying pitch ($p=2d$ to $5d$) is
556 depicted. Except for $s \geq 4d$, it can be observed that at $e \geq 2d$, e has a negligible effect on the
557 joint strength. This value of $2d$ is consistent with the minimum limit of e recommended by
558 [2]. For a constant value of s , the observed increase in strength is related to the increase of the
559 pitch-distance up to $p=3d$. Above this value, the plateau indicates that with the increase of e
560 and p , the joint strength continues to increase with s . This last effect is represented on the
561 negative x-axis for a constant value of $e=3d$. It is shown that the increase of the joint strength
562 is consistent with the increase of the side-distance. From $s=1.5d$ to $3d$, p has no effect on the
563 joint strength. This is because the net-section strength governs this value as shown in Table 6
564 for 1X2 configuration. Above $s=3d$, the effect of p becomes evident while s remains the main
565 parameter affecting the joint strength. FE results presented in Table 6 for 1X2 configuration
566 reveal that the failure mode occurs by bearing for $s > 3d$. The effect of (e) on the joint strength
567 as calculated with [1] shows that limiting the value of s to the minimum values ($s=1.5d$) will
568 limit the strength of the two-bolt joint in a column to that of one-bolt joint with similar width.
569 Similar interpretation can be obtained from FE results. For example in Table 5 for $s=1.5d$,
570 $e=2d$, the load of one-bolt connection is equal to 47.8 kN. In Table 6, for two-bolt in a
571 column, $s=1.5d$, $p=3d$, failure load is equal to 48.4 kN. Therefore, the choice of s should be
572 consistent with the number of bolts in the column and the plate gross area (A_g). For this
573 configuration, $s=3d$ was found sufficient to improve the joint strength to the nominal capacity
574 of the two bolts in shear. This value is lower than the maximum limit recommended by [1],
575 although 9% higher than that of [2]. On the other hand, the ratio of $A_n/A_g = 0.82 < 0.85$ is
576 obtained. With this choice of s , the values of p and e can therefore be limited to $3d$ and $2d$
577 respectively for $t=6.35$ mm and $d=12.7$ mm. The optimum parameters of two-bolt in a

578 column with varying bolt sizes, plate thickness and bolt grade are presented in Table 8. The
579 optimum value of s is found to vary with the plate thickness and the strength of the bolts and
580 should be limited to $4d$ although at this value, $A_n/A_g = 0.86$.

581 8.4. Evaluation of the optimum geometric parameters for two-bolt joint in a row

582 Figure 10(c) presents the effects of joint parameters for two-bolt in a row, as obtained from
583 the FE analysis for $t=6.35$ mm and $d=12.7$ mm. The predicted strengths of the two-bolt joint
584 using the limit values recommended by [1] are also presented. These limit values of g are
585 $2.4d \leq g \leq 7d$ ($14t$). The effect of e presented on the positive x-axis is evaluated by
586 considering a $3d$ side-distance with varying end-distance ($e=1.5d$ to $5d$) and gage-distance
587 ($g=2d$ to $5d$). It can be observed that both g and e have an effect on the joint strength. The
588 effect of e becomes negligible at $4d$ with $g>3d$ while the joint strength continues to increase
589 with the gage-distance. The effect of s with a constant $e=3d$, varying side-distance ($s=1.5d$ to
590 $5d$) and gage-distance ($g=2d$ to $5d$) is shown on the negative x-axis of Figure 10(c). It can be
591 observed that the joint strength increases with s up to $3d$ while it continues to increase with g .
592 At minimum recommended values of s and g , the predicted strength of the connection is
593 governed by net-section failure. At maximum recommended values, bearing is the predicted
594 failure mode. FE results presented in Table 6 for 2X1 configuration show that block-shear
595 and bearing are the observed failure modes for $s \geq 2d$ while net-section failure is observed with
596 narrow side-distance ($s=1.5d$) and shear tear-out failure for short end-distance ($e=1.5d$). With
597 $e=3d$, limiting s and g to the minimum values recommended by [1] ($s=1.2d$ and $g=2.4d$)
598 limits the predicted nominal strength to that of the one-bolt joint with similar width.
599 However, with the same end-distance, increasing s to $3d$ and simultaneously considering
600 $g=3d$ improve the predicted nominal strength to that of the two bolts (A325). With these
601 values, the ratio of $A_n/A_g = 0.78$ which is lower than the limit of 0.85 is obtained. From values
602 reported in Table 6 and the above analysis, $e=3d$, $s=3d$ and $g=3d$ are proposed for $d=12.7$ mm

603 and $t=6.35$ mm. The proposed values are between the limits recommended by [1] although e
604 and s are 9% above the maximum recommended by [2]. The optimum parameters of two-bolt
605 in a row with varying bolt size, plate thickness and bolt grade are presented in Table 9. It is
606 shown that for $s=3d$, the proposed optimums can reach the limit of $g=5d$ and $e=3d$. However,
607 for this value of g , the $A_n/A_g = 0.82$.

608 **9. Conclusion**

609 The aim of this study was to critically examine the recommendations of three design
610 references namely EC9 [1], AASHTO [2] and CSA-S157 [3], to evaluate the maximum
611 geometric parameters beyond which no major increase of joint strength is observed and to
612 propose optimum geometric parameters for one-bolt and two-bolt connections. The chosen
613 optimum was based on the values at which the shear strength of the three grades of bolt
614 (A307, A325, and A490) is reached and on the limit of $A_n \leq 0.85A_g$. It was found that the joint
615 geometric parameters have a high impact on the load performance and failure mode. In
616 summary:

- 617 - For one-bolt configuration, experimental analysis on aluminum-to-steel connections
618 shows that to achieve bearing failure, e and s should not be less than $2.5d$. Above this,
619 bearing damage was observed on steel plate and on the bolt. It was shown from the
620 finite element analysis that maximizing the geometric parameters can result in an
621 increase of the joint strength of more than 75% compared to the minimum values
622 recommended by the design references. Although maximum e and s was observed at
623 $4d$ and $3d$ respectively, the optimum values were found to be related to the bolt
624 diameter, bolt grade and plate thickness. The proposed optimum were within the limit
625 of EC9 [1] and/or AASHTO [2] recommended values and the ratio of $A_n/A_g < 0.85$.
- 626 - For two-bolt connections in a column, it was observed from the experimental study
627 that pure bearing failure is not likely to occur. The pitch $p=3d$ was found sufficient to

628 sustain the joint and prevent the failure between the holes. Finite element analysis
629 indicates that the side-distance is the main parameter that controls the strength of the
630 connection. Limiting the side-distance to the minimum recommended value $s=1.5d$
631 was found to limit the strength of two bolts in a column to that of the one bolt
632 connection. The effect of p was evident after $3d$ while $e>2d$ was found to have limited
633 effect on the joint strength. For this configuration, the proposed optimum parameters
634 should be taken at $e=2d$, $p=3d$ while s varies with the plate thickness and the nominal
635 shear strength of the bolts and should be limited to $4d$ although at this value, $A_n/A_g =$
636 0.86 .

- 637 - For the configuration with two-bolt in a row, the strength of the connection was found
638 to increase with s and e up to the limit value of $3d$ and $4d$ respectively. The increase
639 of the joint strength was consistent with the increase of g . With the proposed
640 optimum, the strength of connection reached the nominal shear strength of the bolts
641 and the ratio of A_n/A_g was less than 0.85 .
- 642 - Compared to double-lap connection, the effect of the out-of-plane deformation
643 observed experimentally in the single-lap configuration with $s=1.5d$ and $e=1.5d$ did
644 not change the failure mode of the aluminum specimen. However, restraining the joint
645 eccentricity with a double-lap configuration was found to slightly improve the joint
646 strength. Out-of-plane deformation was more pronounced in connection with long
647 end-distance and thinner plate.
- 648 - Predicted failure modes were not always consistent with experimental failure mode.
649 In most cases, bearing was found to govern the strength of the connections. The
650 calculated bearing strengths were found to underestimate significantly the connection
651 strength. Compared to EC9 [1] and AASHTO [2], CSA-S157 [3] was found to
652 provide a better prediction of the failure mode and failure load of the connections.

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657

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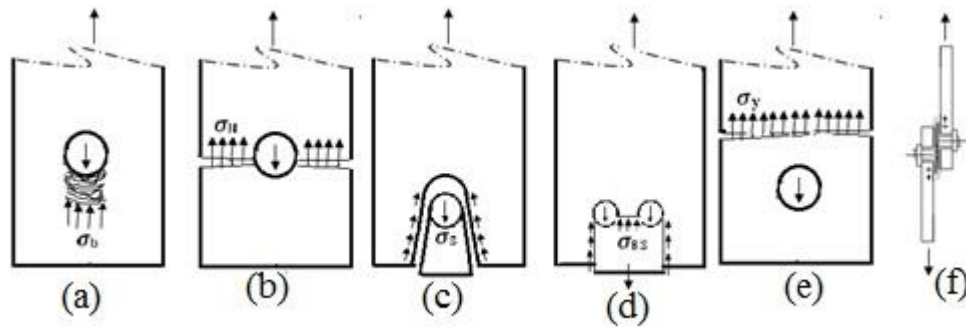
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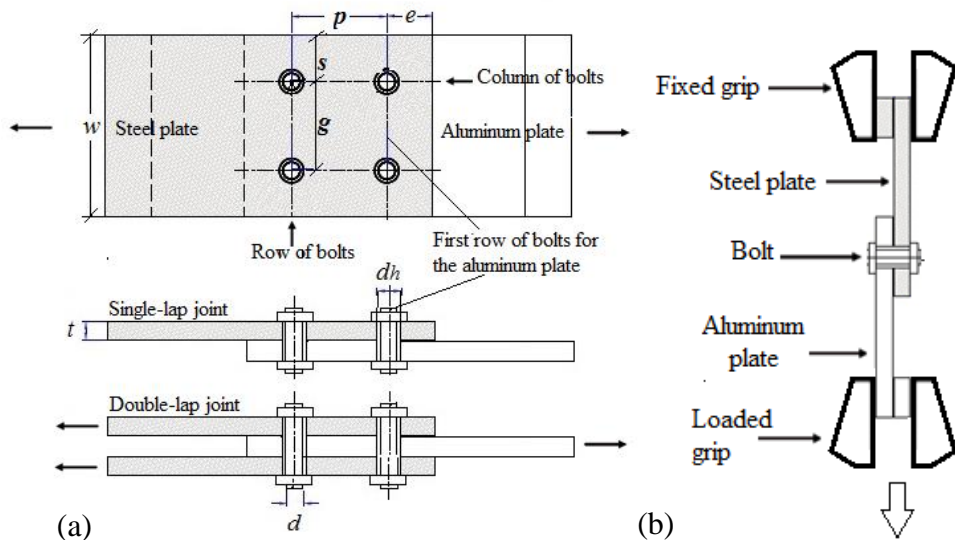


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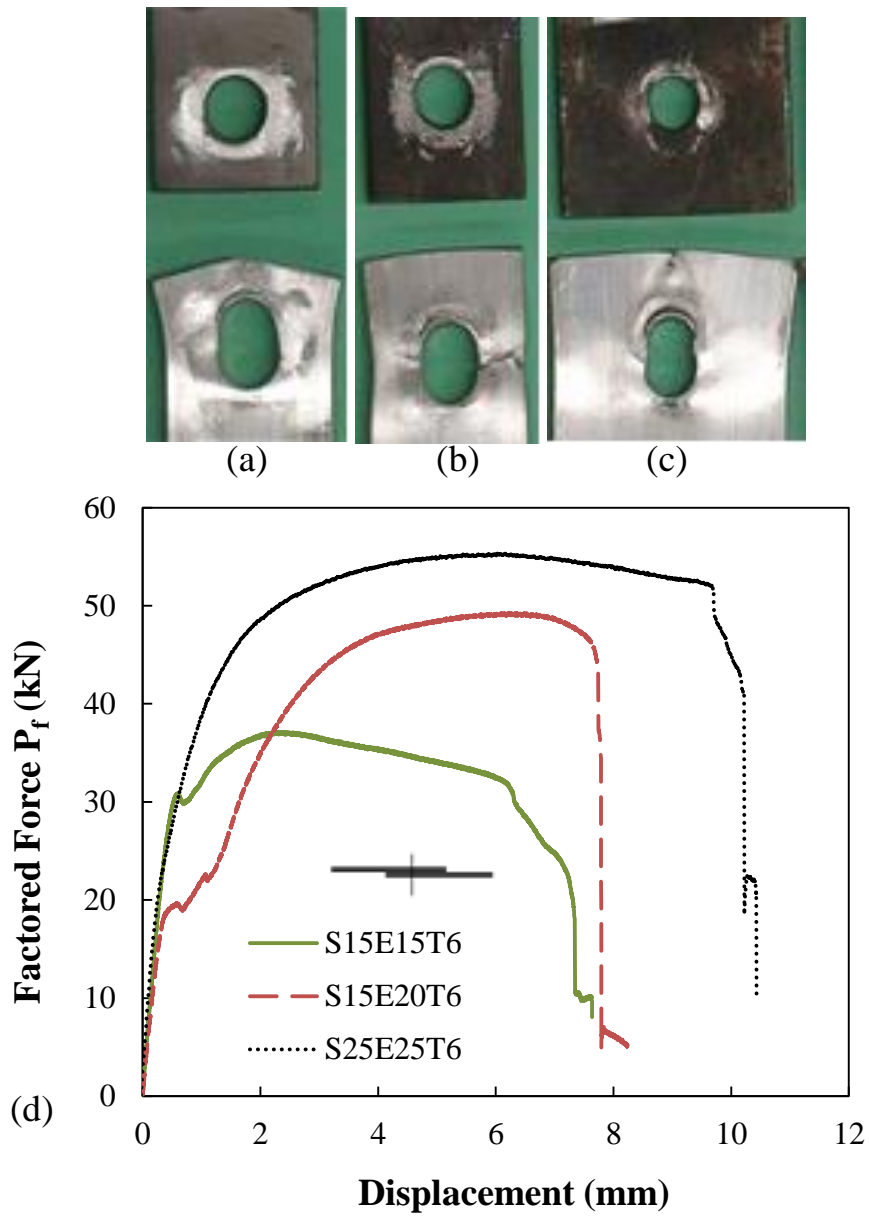
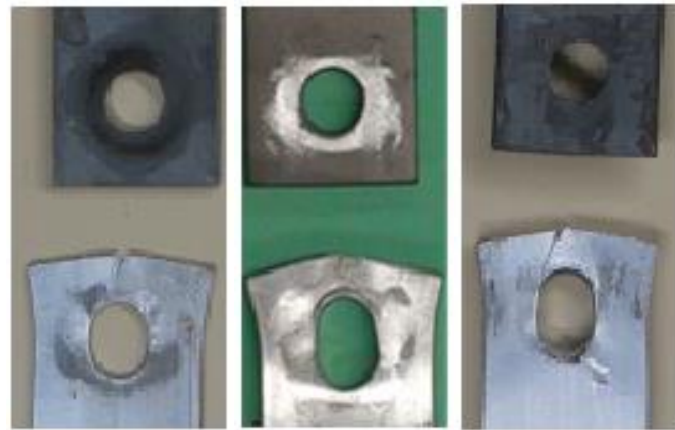


Figure 3. Effect of s and e on one bolt single-lap joints: Damage on: (a) S15E15T6, (b) S15E20T6, (c) S25E25T6, (d) Factored Force-Displacement curves of one bolt single-lap joints.

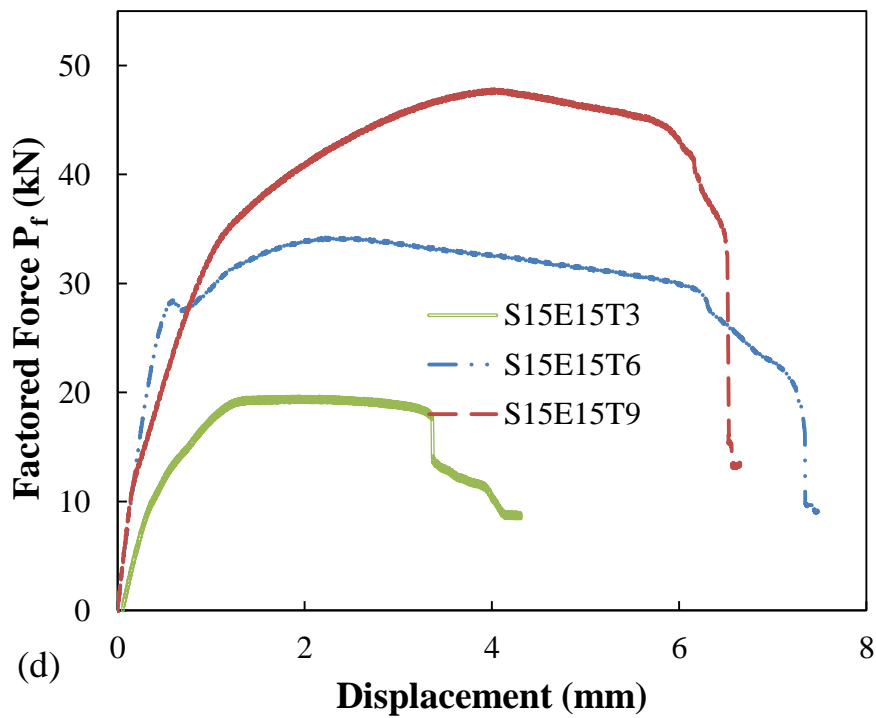
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(a) (b) (c)



(d)

Figure 4. Effect of plate thickness on one bolt single-lap joints: Damage on: (a) S15E15T3; (b) S15E15T6; (c) S15E15T9, (d) Factored Force-Displacement curves

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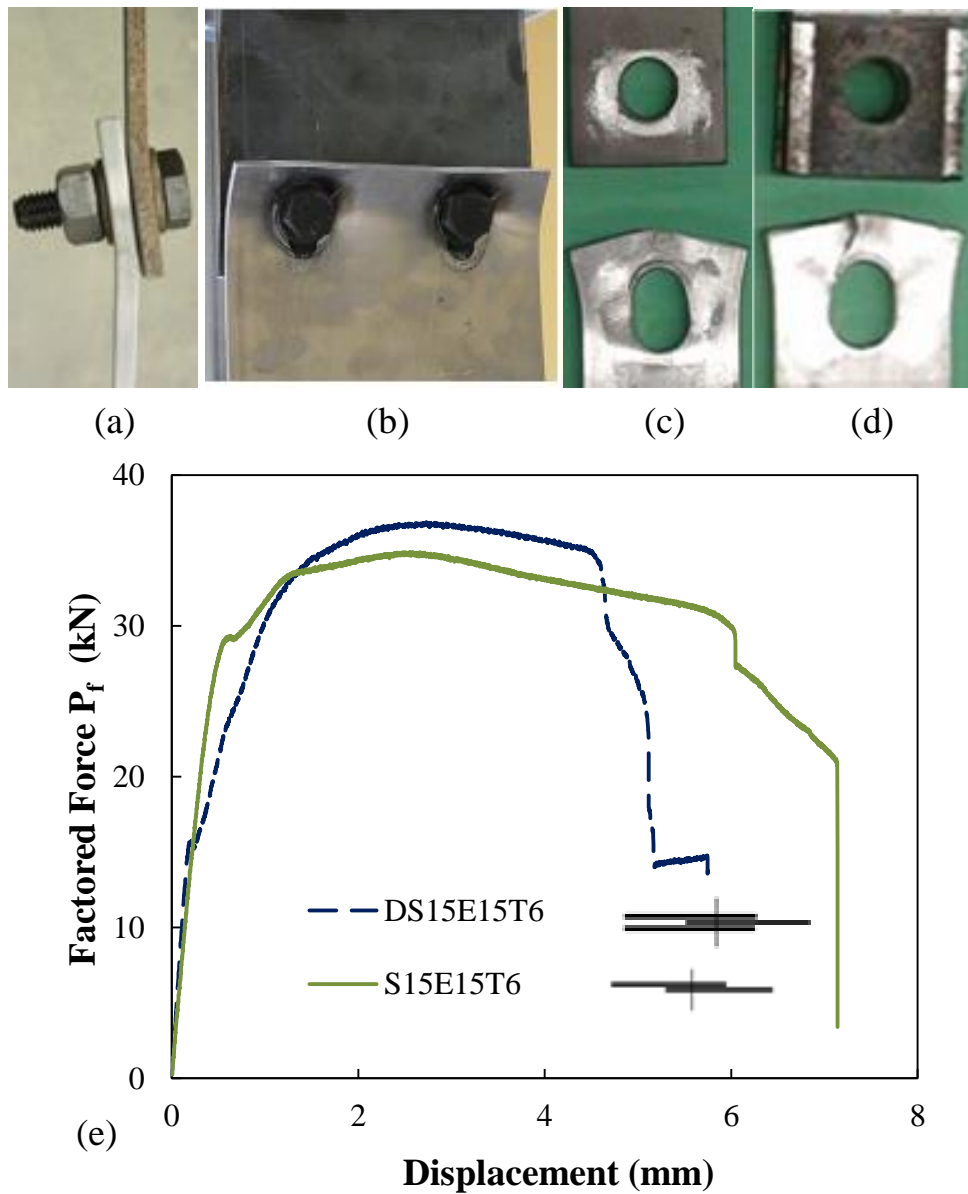


Figure 5. Effect of joint eccentricity: Out-of-plane deformation of (a) S15E15T6 and (b) S30E30G60T3. (c) Fracture of S15E15T6, (d) Fracture of DS15E15T6, (e) Factored force-displacement curves of DS15E15T6 compared to S15E15T6

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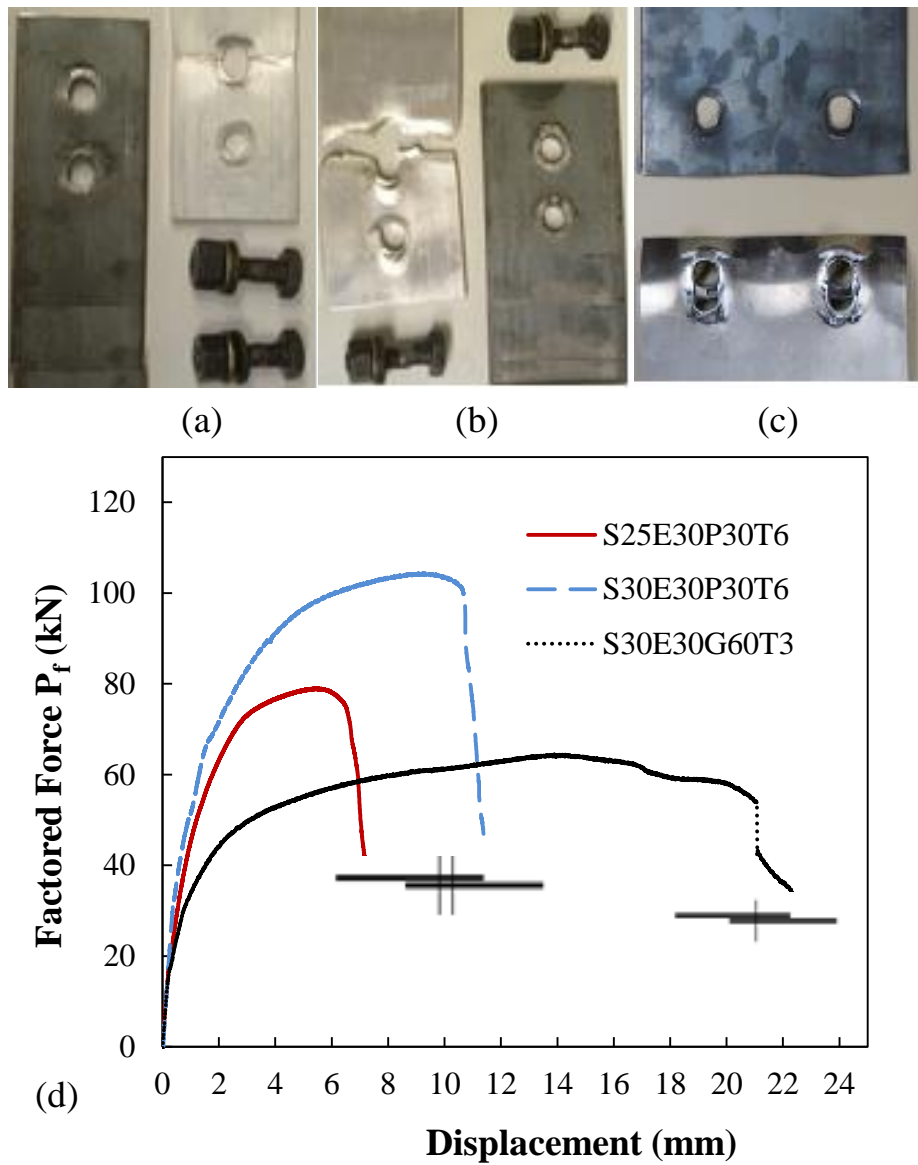


Figure 6. Effect of geometrical parameters in two-bolt single-lap configuration: Damages on 2 bolts joints: (a) S25E30P30T6; (b) S30E30P30T6; (c) S30E30G60T3, (d) Factored force-displacement curves of S30E30P30T6, S25E30P30T6 and S30E30G60T3

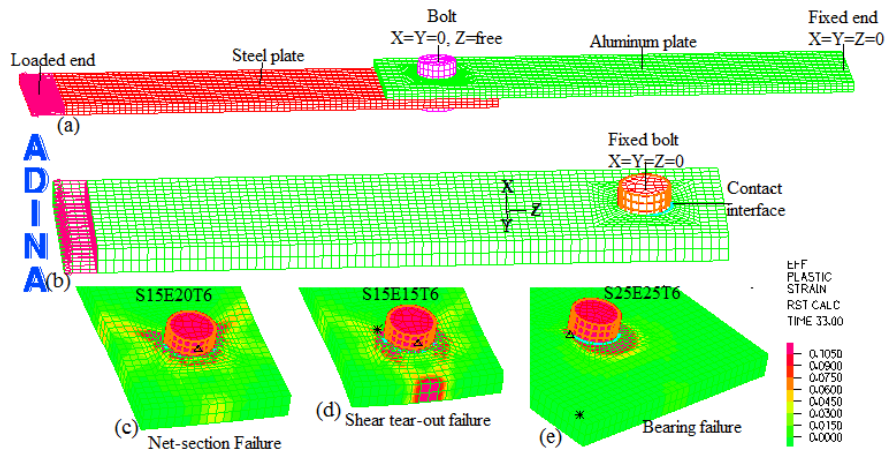


Figure 7. FE models: (a) Typical FE_3DAS model, (b) Typical FE_3DA model. Post-processing response: (c) Net-section failure, (d) Shear tear-out failure, (e) Bearing failure

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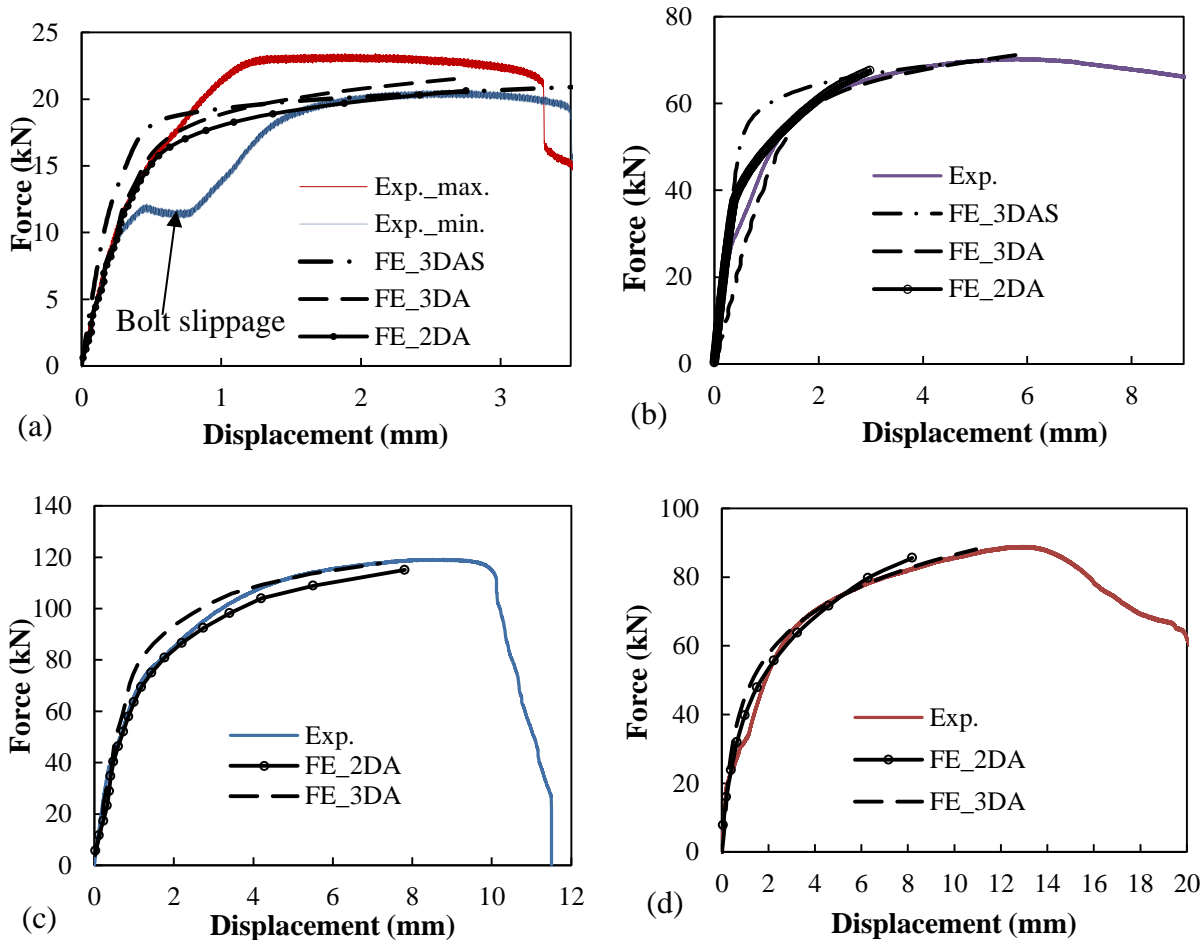


Figure 8. Typical force-displacement curves of the experimental compared to finite element models: (a) S15E15T3, (b) S25E25T6, (c) S30E30P30T6, (d) S30E30G60T3

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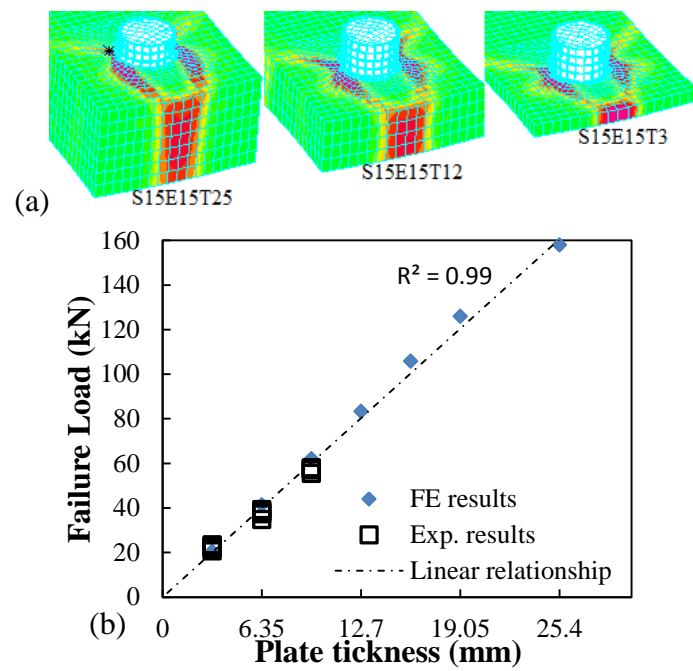


Figure 9. Effect of plate thickness on the joint strength: (a) failure modes, (b) Relationship Failure Load-Plate thickness

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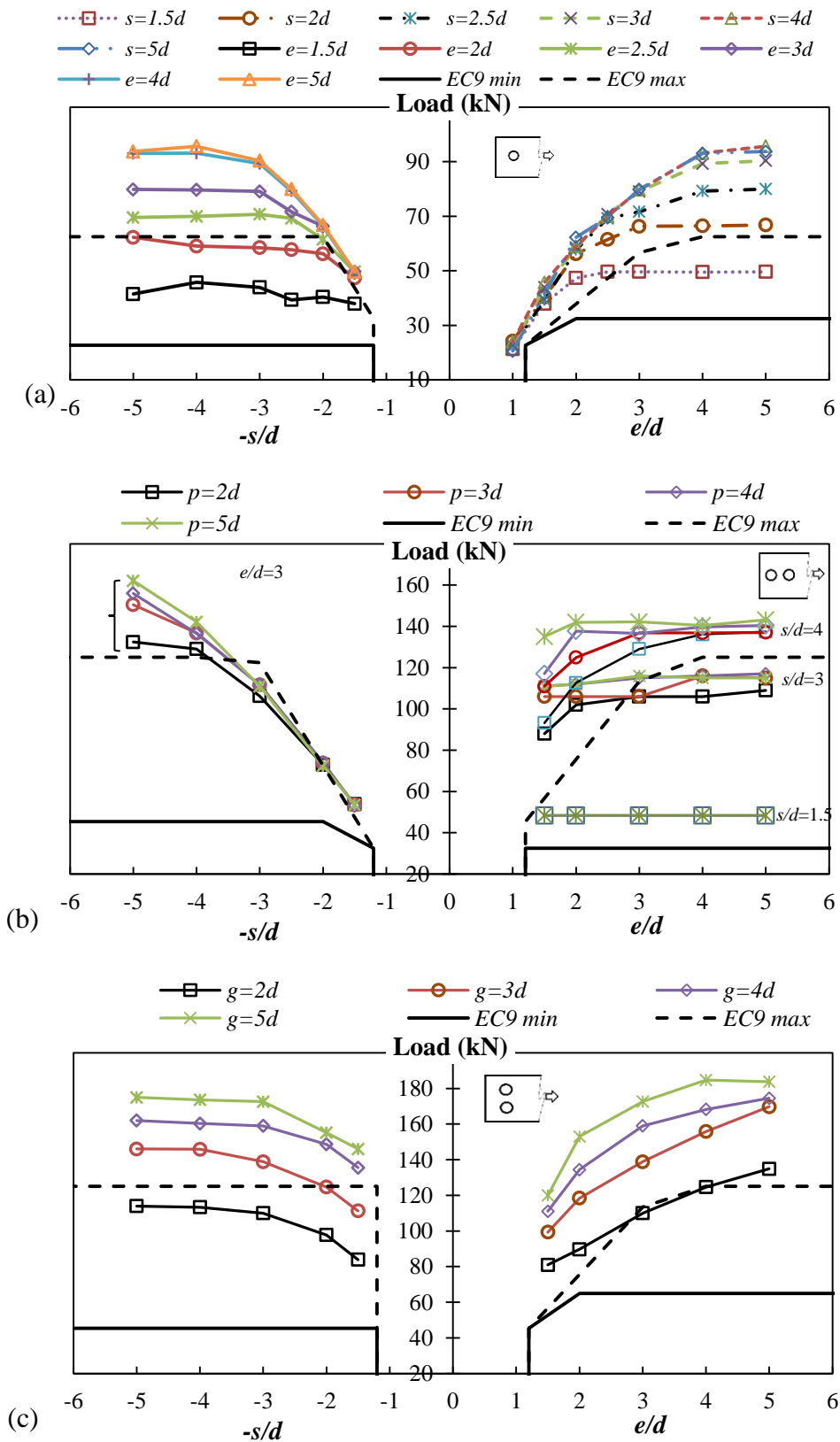


Figure 10. Effect of (a) end-distance and side-distance on one bolt, (b) end-distance and side-distance on 2-bolt in a column, (c) end-distance and side-distance on 2-bolt in a row

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Table 1. Geometric recommendations for aluminum bolted connection

Design Codes	Pitch (p)		Gage (g)		End-distance (e)		Side-distance (s)	
	Min.	Max.	Min.	Max.	Min.	Max.	Min.	Max.
CSA-S157 [3]	$3d$	-	$2.5d$	-	$1.5d$	-	$1.25d$	-
AASHTO [2]	$2.5d$	$17t$	$2.5d$	$17t$	$2d_h$	$5.5t$ or 90 mm	$1.5d_h$	$5.5t$ or 90 mm
EC9 [1]	$2.2d$	$14t$ or 200 mm	$2.4d$	$14t$ or 200 mm	$1.2d$	$4t + 40$ mm	$1.2d$	$4t + 40$ mm

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Table 2. Aluminum tensile test results (coupons)

Corresponding bolted specimens	Coupon name	f_u (MPa)	$f_{0.2}$ (MPa)	ϵ_{fu} (%)	$\epsilon_{0.2}$ (%)	E (GPa)
Al-St S15E15T3	Al _{A,1}	310.4	280.5	-	0.54	78.4
	Al _{A,2}	309.1	277.5	7.7	0.52	69.3
	Al _{A,3}	310.2	279.6	9.6	0.53	67.5
	<i>Aver.</i>	<i>309.9</i>	<i>279.2</i>	<i>8.7</i>	<i>0.53</i>	<i>71.7</i>
Al-St S15E15T6	Al _{B,1}	305.1	279.0	8.0	0.43	69.4
	Al _{B,2}	300.1	-	-	-	-
	<i>Aver.</i>	<i>302.6</i>	-	-	-	-
Al-St S15E15T9	Al _{C,1}	312.2	293.3	7.8	0.62	64.7
	Al _{C,2}	313.7	294.3	8.0	0.6	70.9
	Al _{C,3}	311.3	292.9	7.8	0.61	73.0
	<i>Aver.</i>	<i>312.4</i>	<i>293.5</i>	<i>7.9</i>	<i>0.61</i>	<i>69.5</i>
Al-St S15E20T6	Al _{B,1}	305.1	279.0	8.0	0.43	69.4
	Al _{B,2}	300.1	-	-	-	-
	<i>Aver.</i>	<i>302.6</i>	-	-	-	-
Al-St S25E25T6	Al _{D,1}	330.4	302.7	9.0	0.47	75.5
	Al _{D,2}	331.1	302.6	10.0	0.47	71.4
	<i>Aver.</i>	<i>330.7</i>	<i>302.7</i>	<i>9.5</i>	<i>0.47</i>	<i>73.4</i>
St-Al.-St DS15E15T6	Al _{E,1}	330.1	300.6	9.1	0.46	69.6
	Al _{E,2}	331.1	300.2	9.5	0.46	70.6
	<i>Aver.</i>	<i>330.6</i>	<i>300.4</i>	<i>9.3</i>	<i>0.46</i>	<i>70.1</i>
Al-St S25E30P30 T6	Al _{F,1}	301.8	273.0	9.5	0.42	68.9
	Al _{F,2}	297.8	264.5	9.1	0.41	70.0
	<i>Aver.</i>	<i>299.8</i>	<i>268.8</i>	<i>9.3</i>	<i>0.42</i>	<i>69.4</i>
Al-St S30E30P30 T6	Al _{G,1}	310.6	273.0	8.6	0.42	63.1
	Al _{G,2}	312.8	275.0	10.2	0.42	71.3
	<i>Aver.</i>	<i>311.7</i>	<i>274.0</i>	<i>9.4</i>	<i>0.42</i>	<i>67.2</i>
Al-St S30E30G6 T3	Al _{H,1}	359.5	318.0	10.3	0.53	66.3
	Al _{H,2}	358.1	315.0	9.3	0.5	76.6
	Al _{H,3}	351.5	309.8	12	0.52	70.1
	<i>Aver.</i>	<i>356.4</i>	<i>314.3</i>	<i>10.5</i>	<i>0.52</i>	<i>71.0</i>
Design references	CSA-S157	260	240	-	-	70
	AASHTO	290	240	10	-	70
	EC9	290	240	-	-	70

Al: aluminum, St: steel

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Table 3. Experimental test results compared to predicted results

Test names	Experimental (aver . load)				EC9 (2007)			AASHTO (2007)			CSA-S157 (2007)		
	1	2	3	4	5	6	7	8	9	10	11	12	13
	P_{exp} (kN)	Std. dev.	P_f (kN)	FM	P_n (kN)	1/5	FM	P_n (kN)	1/8	FM	P_n (kN)	1/11	FM
S15E15T3	22.4	1.09	18.8	S	14.2	1.58	B	14.0	1.60	B	18.7	1.20	S
S15E15T6	39.6	0.66	33.8	S	27.7	1.43	B	28.0	1.41	B	36.6	1.08	S
S15E15T9	57.1	1.04	47.5	S	43.1	1.32	B	44.1	1.29	B	56.7	1.01	S
S15E20T6	52.6	0.56	44.8	N	36.9	1.43	B	37.4	1.41	B	46.3	1.14	N
S25E25T6	69.4	0.68	54.4	B	50.4	1.38	B	40.6	1.71	B	53.3	1.30	B
DS15E15T6	45.6	1.51	35.9	S	30.2	1.51	B	30.1	1.51	B	40.0	1.14	S
S25E30P30T6	95.0	0.83	79.3	N	79.5	1.19	B	72.0	1.32	B	96.7	0.98	N
S30E30P30T6	120.2	0.63	104.1	N	82.7	1.45	B	73.4	1.64	B	100.3	1.20	B
S30E30G60T3	86.84	1.13	63.4	B	65.2	1.33	B	42.08	2.06	B	57.5	1.51	B

FM: Failure mode, N: net-section failure; B: bearing failure, S: shear tear-out failure

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Table 4. Experimental versus FE predicted results

	P_{exp} (kN)	Exp. FM	3D P_{FE} (kN)	FE FM	P_{exp}/P_{3DFE}	2D P_{FE} (kN)	P_{exp}/P_{2DFE}
S15E15T3	22.4	S	21.6	S	1.04	20.6	1.09
S15E15T6	39.6	S	39.7	S	0.99	38.0	1.04
S15E15T9	57.1	S	56.4	S	1.01	58.7	0.97
S15E20T6	52.6	N	52.9	N	0.99	47.3	1.11
S25E25T6	69.4	B	71.5	B	0.97	69.8	0.99
DS15E15T6	45.6	S	42.3	S	1.08	41.4	1.10
S25E30P30T6	95.0	N	98.3	N	0.97	102.8	0.92
S30E30P30T6	120.2	N	117.6	N	1.02	115.1	1.04
S30E30G60T3	86.84	B	88.3	B	0.98	85.6	1.01
				Average	1.01		1.03
				Standard deviation	0.04		0.06

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Table 5. FE results for one-bolt connection ($t=6.35$ mm and $d=12.7$ mm)

One-bolt (1X1) failure load (kN)/failure mode					
s/d	e/d				
	1.5	2	3	4	5
1.5	41.4/S	47.3/N	49.6/N	49.5/N	49.6/N
2	45.7/S	56.2/S	66.3/N	66.5/N	66.8/N
3	43.9/S	58.5/S	79.1/B	89.3/B	90.4/B
4	39.3/S	59.0/S	79.7/B	93.1/B	95.7/B
5	40.4/S	62.2/S	79.8/B	93.0/B	93.8/B

N: net-section failure; B: bearing failure, S: shear tear-out failure

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Table 6. FE results for two-bolt connection ($t=6.35$ mm and $d=12.7$ mm)

Two-bolt in a column (1X2) FL (kN)/FM						Two-bolt in a row (2X1) FL (kN)/FM					
s/d	e/d	p/d				s/d	e/d	g/d			
		2	3	4	5			2	3	4	5
1.5	1.5	48.4/N	48.4/N	48.4/N	48.4/N	1.5	1.5	69.0/S	103/S	101/S	96.7/S
	2	48.4/N	48.4/N	48.4/N	48.4/N		2	79.1/N	109N	130/K	133/K
	3	48.4/N	48.4/N	48.4/N	48.4/N		3	83.9/N	111/N	136/K	152/K
	4	48.4/N	48.4/N	48.4/N	48.4/N		4	84.9/N	112/N	140/N	148/N
	5	48.4/N	48.4/N	48.4/N	48.4/N		5	85.7/N	111/N	140/N	160/N
2	1.5	71.5/N	73.0/N	73.0/N	73.0/N	2	1.5	76.5/S	114/S	109/S	105/S
	2	72.4/N	73.5/N	72.7/N	72.1/N		2	75.9/K	108/K	123/B	122/B
	3	72.9/N	73.8/N	74.0/N	72.4/N		3	91.7/K	125/K	145/B	155/B
	4	73.2/N	73.7/N	72.9/N	72.8/N		4	91.2/K	126/K	148/B	168/B
	5	73.0/N	73.5/N	72.7/N	72.2/N		5	92.0/K	128/K	147/B	168/B
3	1.5	88/N	106/N	111/N	111/N	3	1.5	81.0/S	99.3/S	111/S	120/S
	2	102/N	106/N	112/N	112/N		2	89.7/K	118/K	134/B	153/B
	3	106/N	115/N	115/N	116/N		3	110/K	139/K	159/B	173/B
	4	106/N	116/N	116/N	115/N		4	125/K	156/K	168/B	185/B
	5	109/N	115/N	117/N	115/N		5	135/K	170/K	175/B	184/B
4	1.5	93.2/B	111/B	117/B	135/B	4	1.5	85.6/S	112/S	113/S	115/S
	2	113/B	125/B	138/B	142/B		2	87.2/K	128/K	139/B	133/B
	3	129/B	137/B	137/B	142/B		3	113/K	146/K	160/B	174/B
	4	136/B	137/B	140/B	141/B		4	118/K	154/K	185/B	177/B
	5	137/B	137/B	141/B	143/B		5	123/K	166/K	189/B	172/B
5	1.5	90.7/B	105/B	117/B	115/B	5	1.5	85/S	115/S	114/S	117/S
	2	112/B	133/B	143/B	164/B		2	81.7/K	115/K	130/B	127/B
	3	132/B	151/B	156/B	162/B		3	119/K	146/K	162/B	174/B
	4	154/B	158/B	165/B	180/B		4	128/K	158/K	177/B	176/B
	5	151/B	167/B	165/B	162/B		5	133/K	164/K	180/B	185/B

FL: failure load; FM: failure mode; N: net-section failure; B: bearing failure; S: shear tear-out failure; K: block shear failure

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Table 7. Optimum parameters for one-bolt connection

Bolt grade	t (mm)	$d=12.7$ mm		$d=15.88$ mm		$d=19.05$ mm		$d=25.4$ mm	
		e/d	s/d	e/d	s/d	e/d	s/d	e/d	s/d
A307	6.35	1.5	1.5	2	1.5	2	1.5	2.5	2.5
	9.53	1.5	1.5	1.5	1.5	1.5	1.5	2	1.5
	12.7	1.5	1.5	1.5	1.5	1.5	1.5	1.5	1.5
	25.4	1.5	1.5	1.5	1.5	1.5	1.5	1.5	1.5
A325	6.35	3	2	3.5	2.5	-	-	-	-
	9.53	2	1.5	2	2	2.5	2.5	4	3
	12.7	1.5	1.5	2	1.5	2	1.5	2.5	2.5
	15.88	1.5	1.5	1.5	1.5	1.5	1.5	2	2
	19.05	1.5	1.5	1.5	1.5	1.5	1.5	2	1.5
	25.4	1.5	1.5	1.5	1.5	1.5	1.5	1.5	1.5
A490	6.35	3	3	-	-	-	-	-	-
	9.53	2	2	3	2	3.5	2.5	-	-
	12.7	1.5	2	2	2	2.5	2	3.5	2.5
	15.88	1.5	1.5	2	1.5	2	1.5	3	2
	19.05	1.5	1.5	1.5	1.5	2	1.5	2	2
	22.22	1.5	1.5	1.5	1.5	1.5	1.5	2	1.5
	25.4	1.5	1.5	1.5	1.5	1.5	1.5	2	1.5

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Table 8. Optimum parameters for two-bolt in a column ($e=2d$)

Bolt grade	t (mm)	$d=12.7$ mm		$d=15.88$ mm		$d=19.05$ mm		$d=25.4$ mm	
		p/d	s/d	p/d	s/d	p/d	s/d	p/d	s/d
A307	6.4	2	2	2	3	2	3	2	4
	9.5	2	1.5	2	1.5	2	2	2	3
	12.7	2	1.5	2	1.5	2	1.5	2	2
	25.4	2	1.5	2	1.5	2	1.5	2	1.5
A325	6.4	3	3	-	-	-	-	-	-
	9.5	2	3	2	3	2	4	-	-
	12.7	2	2	2	3	2	3	2	4
	15.9	2	1.5	2	2	2	3	2	3
	19.1	2	1.5	2	1.5	2	2	2	3
	25.4	2	1.5	2	1.5	2	1.5	2	2
A490	9.5	2	3	3	4	-	-	-	-
	12.7	2	3	2	3	2	4	-	-
	15.9	2	2	2	3	2	3	2	4
	19.1	2	1.5	2	2	2	3	2	3
	22.2	2	1.5	2	2	2	2	2	3
	25.4	2	1.5	2	1.5	2	2	2	3

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Table 9. Optimum parameters for two-bolt in row ($s=3d$)

Bolt grade	t (mm)	$d=12.7$ mm		$d=15.88$ mm		$d=19.05$ mm		$d=25.4$ mm	
		g/d	e/d	g/d	e/d	g/d	e/d	g/d	e/d
A307	6.4	2	1.5	2	1.5	3	1.5	3	3
	9.5	2	1.5	2	1.5	2	1.5	2	2
	12.7	2	1.5	2	1.5	2	1.5	2	1.5
	25.4	2	1.5	2	1.5	2	1.5	2	1.5
A325	6.4	3	2.75	4	3	-	-	-	-
	9.5	3	1.5	3	2	3	3	5	3
	12.7	2	1.5	2	1.5	3	1.5	3	3
	15.9	2	1.5	2	1.5	2	1.5	3	2
	19.1	2	1.5	2	1.5	2	1.5	3	1.5
A490	6.4	4	3	-	-	-	-	-	-
	9.5	4	1.5	4	2	4	3	-	-
	12.7	2	1.5	3	3	3	2	4	3
	15.9	2	1.5	3	1.5	3	1.5	3	3
	19.1	2	1.5	2	1.5	2	1.5	3	2
	22.2	2	1.5	2	1.5	2	1.5	3	1.5

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