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NSF-STU Workshop
on
Adaptive Control

9 - 11 JULY 1984

DEPARTMENT OF AUTOMATIC CONTROL
LUND INSTITUTE OF TECHNOLOGY
APRIL 1985

NSF-STU Workshop on Adaptive Control

**Department of Automatic Control
Lund Institute of Technology
Lund Sweden**

9 - 11 July 1984

Edited by Tore Hägglund

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Title and subtitle NSF-STU Workshop on Adaptive Control			
Abstract The National Science Foundation (NSF) and the Swedish Board for Technical Development (STU) have signed an agreement for cooperation. Both agencies are currently supporting research in adaptive control. Within this framework, a workshop was held at the Department of Automatic Control at the Lund Institute of Technology, Lund, Sweden, on July 9-11 1984. This report contains abstracts, copies of the viewgraphs and a summary of the discussions.			
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Introduction

The National Science Foundation (NSF) and the Swedish Board for Technical Development (STU) have signed an agreement for cooperation. Both agencies are currently supporting research in adaptive control. The purpose of the seminar was to bring the research in the supported projects together to provide a perspective of the field, its accomplishments and deficiencies and to find directions for future research.

The laboratory for Information and Decision Systems at the Massachusetts Institute of Technology, directed by Professor Sanjoy Mitter, and the Department of Automatic Control at Lund Institute of Technology, Lund, Sweden, directed by Professor Karl Johan Åström, have long maintained close informal contact on research problems of common interest. There is interest on both sides in formalizing this arrangement so that exchanges of faculty, staff (and possibly students) can take place. The workshop on Adaptive Control was organized within this framework.

The workshop program was discussed with Dr. Abraham Haddad, former Program Director of the Systems Theory and Operations Research Program of the National Science Foundation and Mr Ove Berkefelt Program Director for at the Swedish Board of Technical Development. They were in agreement with the aims and objectives of the workshop.

The workshop was held at the Department of Automatic Control at the Lund Institute of Technology, Lund, Sweden, on July 9 - 11, 1984 this week followed the IFAC Congress in Budapest.

The workshop was informal. There were 14 US and 20 Swedish participants. The formal presentations covered industrial needs and experiences, applications, stability theory, system identification, stochastic adaptive control, unmodeled dynamics and new directions as well as many informal discussions. The workshop was viewed very favourable from the participants who found it a stimulating intellectual experience.

A brief assessment of the field can be summarized as follows. There has been research in adaptive control for at least 30 years. The field is, however, still not in very good shape. There is a proliferation of ideas and techniques, and a lack of coherence. Recently there has, however, been some limited theoretical results. Adaptive techniques are also starting to be used in microprocessor based controllers. It appears as a good research field because theoretical results are badly needed to get insight, to structure the problem, and to unify the field. There is also a considerable industrial interest to use adaptive techniques in many different fields. Several new products have recently been announced. There are several strong research groups in the field, both in the United States and in Europe. The theoretical aspects have been emphasized in U.S. research. In Europe the theoretical research has however also been blended with practice. There are new application areas emerging, e.g. in robotics.

This report contains abstracts, copies of the viewgraphs and a summary of the discussions.

Dedication

This report is dedicated to the memory of Dr. Howard Elliott a prominent researcher in adaptive control. He contributed significantly to the success of the workshop which was the last formal meeting in which he participated.

Program

MONDAY - APPLICATIONS

9.00 Introduction and Welcome

Berkefelt

Cooperation between NSF and STU

9 - 12 Industrial Products

Egardt

The ASEA-Novatune system

Bengtsson

Experiences with the ASEA-Novatune

Åström

Automatic tuning of simple regulators

Bååth

The NAF - Autotuner

13 - 15 Applications

Stein

History and issues in adaptive flight control

Olsson, Rundqwist

Self-tuning control of dissolved oxygen concentration in activated sludge systems

Elliott

Adaptive pole placement for robots and servomechanisms

15 - 17 Discussion

Where do we stand with respect to applications? What algorithms are being used? What things work? What are the difficulties? What tricks are used?

Sternby

Some desirable features of industrial adaptive controllers

TUESDAY - THEORY

8 - 10 Stability Theory

Wittenmark

Self-tuning regulator with increased prediction horizon

Morse

A universal control capable of stabilizing any single-input, single-output, minimum phase linear system of relative degree ≤ 2

Byrnes

Adaptive stabilization of linear multivariable systems

Johansson

Lyapunov functions, cost functions and adaptive control

10 - 12 System Identification

Söderström

Instrumental variable methods for systems operating in closed loop with application to adaptive control

Solo

Adaptive spectral factorization

Ljung

Frequency domain properties of identified transfer functions

13 - 15 Stochastic Adaptive Control

Varaya

Multi-armed bandits

Kumar

On self-tuning to the optimal controller

Millnert

A comparison of some control strategies for systems with fast parameter variations

Hägglund

Recursive estimation of slowly time-varying parameters

15 - 17 Discussion

Where does the theory stand? What results are needed? Do the results cover the problems brought up by practice? What problems can we hope to solve?

WEDNESDAY - ROBUSTNESS AND NEW DIRECTIONS

8 - 11 Unmodeled Dynamics and Robustness

Kokotovic

Robustness of (MRAS) adaptive control

Sastry

Parameter convergence in model reference adaptive control and its impact on robustness

Trulsson

On adaptive control with prescribed robustness properties

Rohrs

On living with the positive real condition

Bertsekas

Distributed asynchronous algorithms for deterministic and stochastic optimization

11 - 12 Demonstration of Control Laboratory

13 - 15 New Directions

Åström

Expert Control

Årzén

Experiments with Expert Control

15 - 17 Discussion

Future directions.

List of Participants

Gunnar Bengtsson	ASEA
Ove Berkefelt	STU
Dimitri Bertsekas	MIT
Torsten Bohlin	KTH
Christopher Byrnes	Harvard
Lars Bååth	NAF Controls
Bo Egardt	ASEA
Howard Elliott	UMass
Gene Franklin	Stanford
Ove Gradin	KTH
Ivar Gustavsson	ASEA
Per Hagander	LTH
Tore Hägglund	LTH
Rolf Johansson	LTH
Petar V Kokotovic	Illinois
P R Kumar	Maryland
Lennart Ljung	LiTH
Mille Millnert	LiTH
Sanjoy K Mitter	MIT
Stephen Morse	Yale
Gustaf Olsson	LTH
Charles Rohrs	Notre Dame
Lars Rundqwist	LTH
Shankar Sastry	Berkeley
Victor Solo	Purdue
Gunter Stein	MIT
Jan Sternby	Gambro
Torsten Söderström	Uppsala
Eva Trulsson	LiTH
Pravin Varaya	Berkeley
Björn Wittenmark	LTH
George Zames	McGill
Karl-Erik Årzén	LTH
Karl Johan Åström	LTH

Technical secretaries

Anders Ahlén	Uppsala
Jan Peter Axelsson	LTH
Mats Lilja	LTH
Bengt Mårtensson	LTH
Per Persson	LTH
Mikael Sternad	Uppsala
Hussein Youlal	LTH
Lars Pernebo	Alfa Laval
Ulf Hagberg	Alfa Laval

Cooperation between NSF and STU

Ove Berkefelt

STU
Stockholm

Nearly three years ago NSF and STU reached a general agreement on cooperation in research. This agreement covers all scientific areas where STU and NSF are funding research projects.

For a small country like Sweden with limited resources in personnel and money it is of course difficult to find areas where our research is of adequate level and size compared to US. I am therefore very pleased to note that we have recently managed to arrange workshops in two areas within the field of electronics, computers and systems sciences. About two months ago we had a joint NSF-STU workshop on computer based vision and this week we will have this workshop on adaptive control.

It is interesting to compare these two areas. Computer based vision is a new science. By a generous budget we have managed to create a good scientific level in a short time, approximately 5 years.

Adaptive control on the other side has been built up gradually during a long time and I would say more thanks to excellent and devoted researchers than to a generous budget. In any case we have in adaptive control enough research results, enough researchers and enough applications to attract the interest from NSF and US researchers and this is in a time when, as I understand it, NSF is putting heavier conditions on research cooperation with other countries than earlier.

The purpose of this workshop is apart from the exchange of research results between the two countries to find out whether adaptive control is an area where we can find possibilities for future cooperation. A joint US-Swedish project on adaptive control would have a great chance to get funding at least from STU. Unfortunately my counterpart from NSF is not here so that we can hear NSF's opinion.

This workshop happens to be very suitably located in time. We are at present at STU planning a new national program in information technology. We are trying to establish which areas in systems and computer sciences where we can compare with other countries and where scientific results are likely to lead to industrial applications and progress. I hope this workshop will help in this respect.

Let me end by saying that I hope that our American guests will have a pleasant time at Lund for some days and that this workshop will lead to deeper contacts between Swedish and US researchers and to a future more or less formalised cooperation.

The ASEA-Novatune system

Bo Egardt

ASEA AB
Västerås, Sweden

Egardt gave an overview of the system hardware, containing process interfaces etc.

The application program is written in a block oriented language. This is the industrial control engineers look at processes and control. Besides the selftuning regulator the language contains arithmetics, logics and other functions like PID.

The signal types in the language are integer and real, and there are modules like selectors, arithmetic operations, logic, delays, interface modules, filters, regulators etc.

All modules are available in the system library. The program is entered via a simple hand terminal or a standard terminal in a laboratory or at the installation site.

The programmer selects sampling intervals and priorities for the different control tasks. Except standard clock interrupt it's possible to use software interrupts or pulse counters to determine the sampling instants.

The system contains three different adaptive regulators built around the same algorithm. The difference between the regulators is the degree of flexibility offered to the user. The regulators are

STAR1	Basic, least complex regulator
STAR2	Medium complex regulator
STAR3	Complex regulator

STAR3 is the most frequently used regulator. STAR2 and STAR3 both contain feedforward, but in STAR3 the number of parameters in the control law is selected by the user.

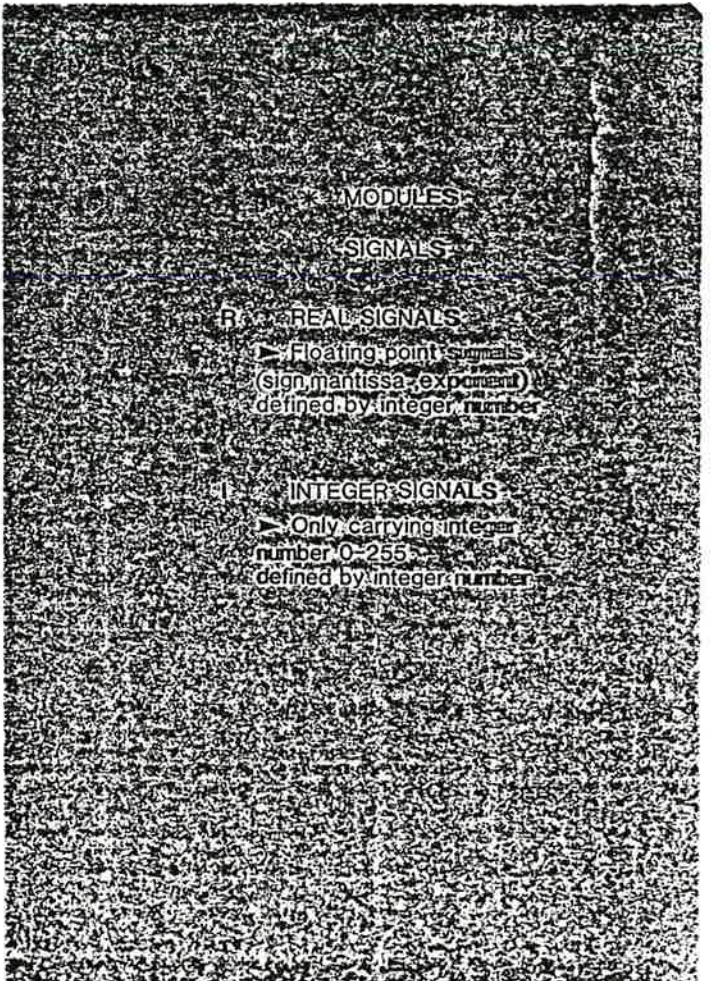
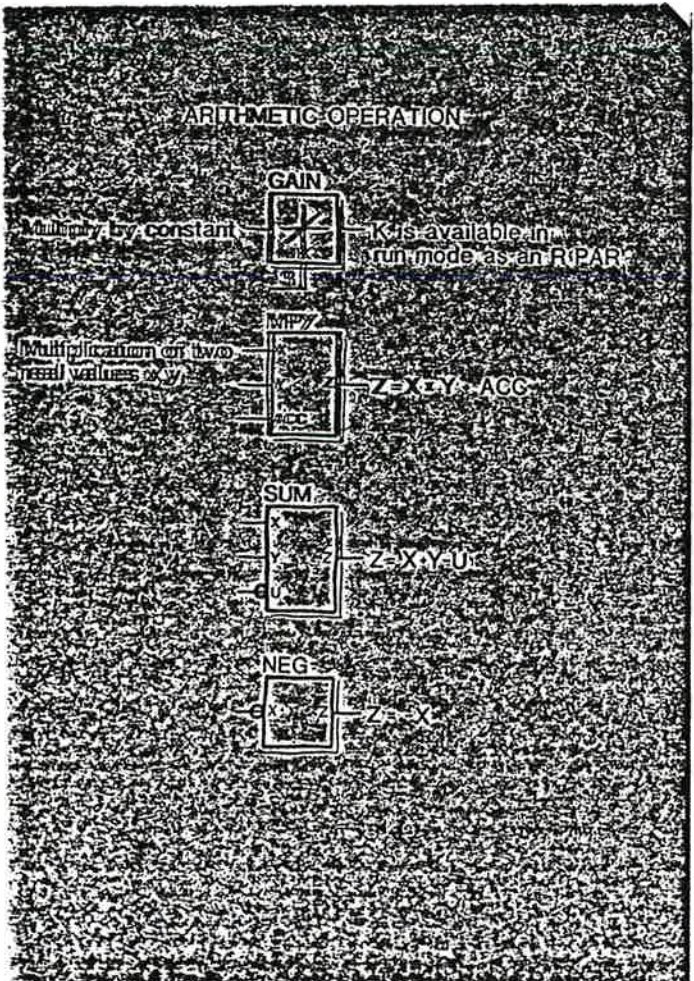
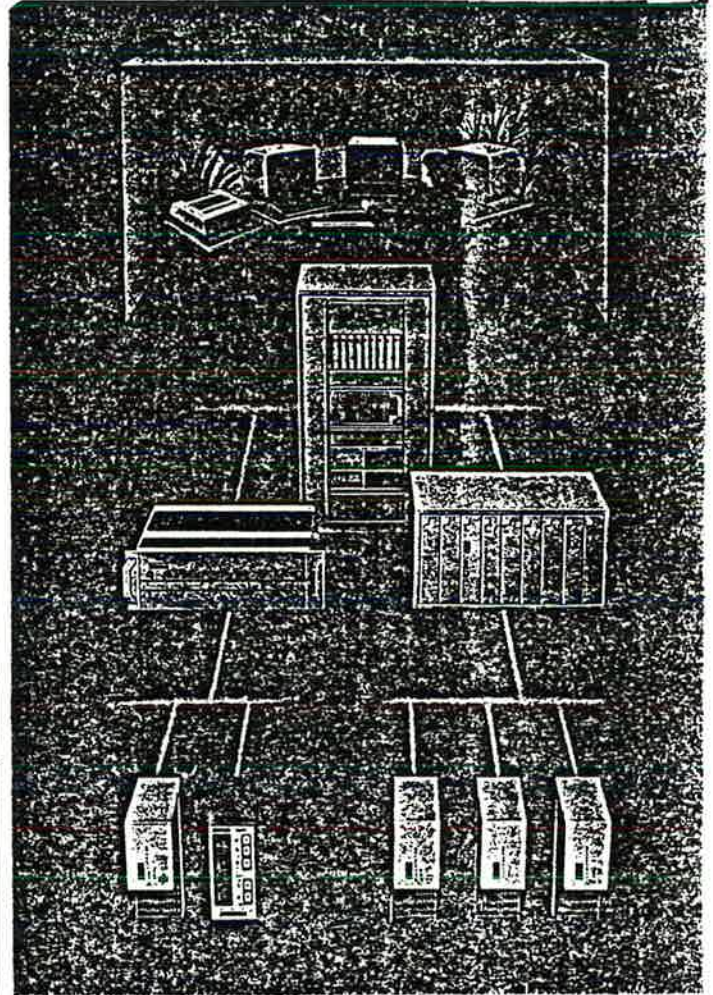
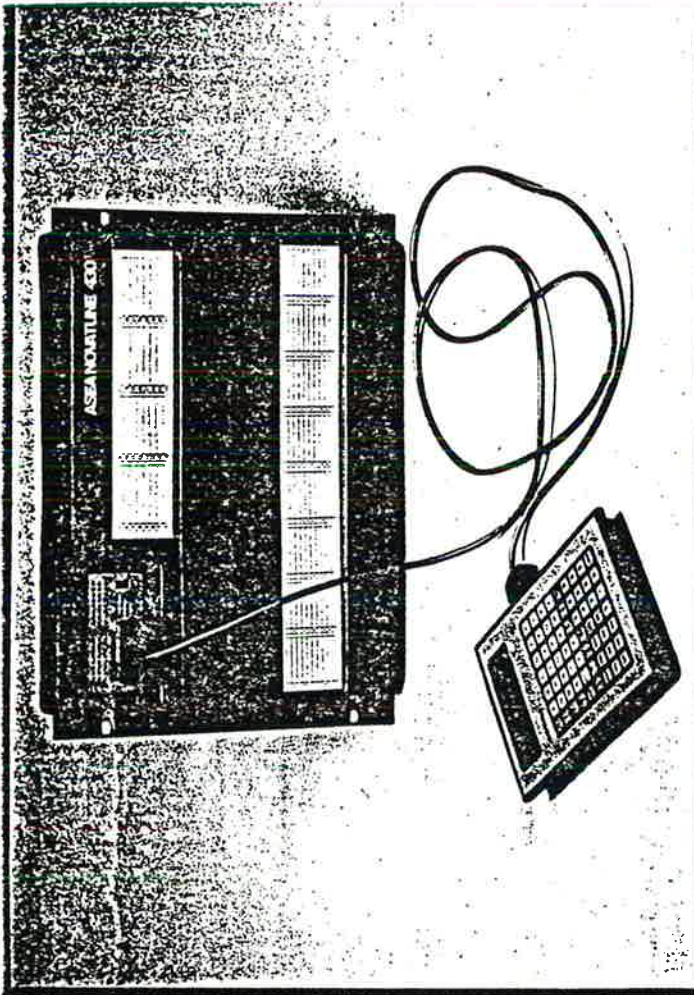
The signals connected to the STAR3 module are the manual control signal, the feedback signal (process output), the reference value and the feedforward signal. The control value can be given both absolute and incremental limits.

Integer signals determine the mode of the regulator, i.e. adaptation can be switched off, saved parameters can be restored etc. Unconnected inputs are given default values.

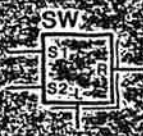
The regulator uses a minimum variance algorithm with least squares identification. Besides, one closed loop pole can be defined, the prediction horizon and the sampling interval can be chosen, and the integral action can be switched on and off. A penalty can be introduced on the control output.

Reference:

Bengtsson G, Egardt B (1984): Experiences with Self-Tuning Control in the Process Industry. Preprints of IFAC 9th World Congress, Budapest, Hungary, vol XI, 132-140.



SELECTORS



LOGICAL MODULES



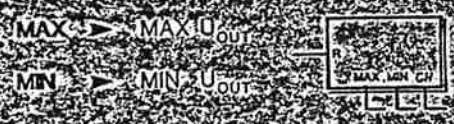
IF S → S_K IS I → S_{K-1}



INTERFACE MODULES



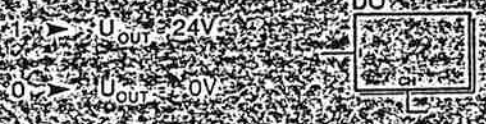
CHANNEL 0-15, 16-31 ETC.



CHANNEL 0-3, 4-7 ETC.



CHANNEL 0-15, 16-31 ETC.

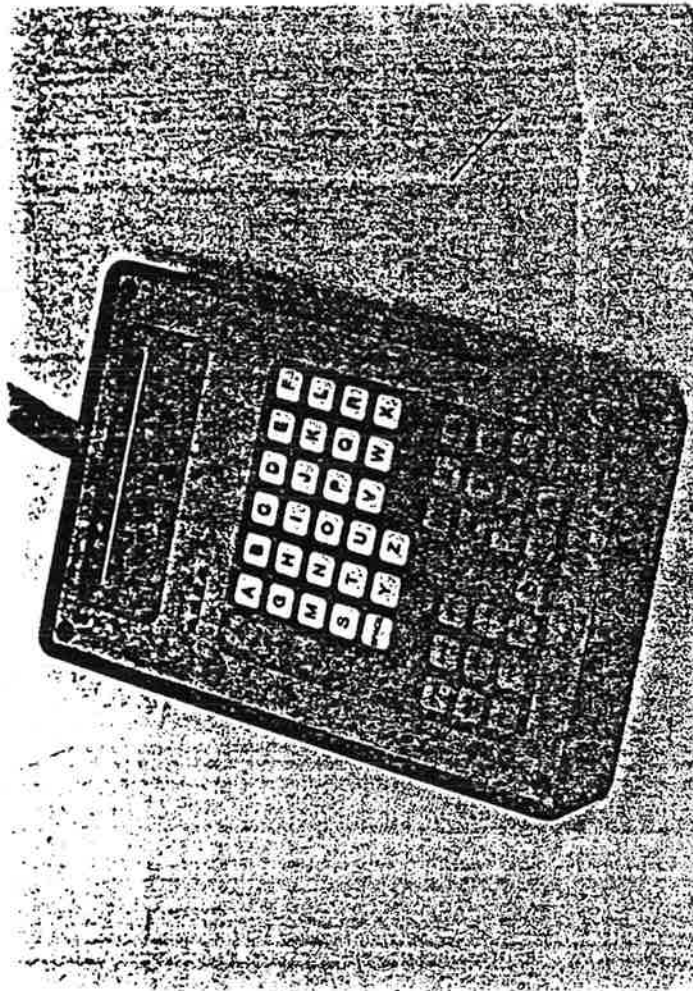


CONTROL MODULES


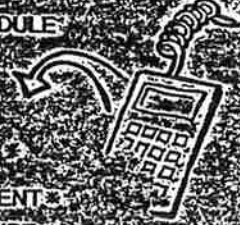


EX: If $t = 1$ sec and the time based is 50 ms
 $T = 0.05$

If on is not connected the filter is on
 *K and $\frac{+}{-}$ are available in run mode.



DEFINING A MODULE

<p>NEW</p> <p>① 1</p> <p>AO</p> <p>② 12</p> <p>5</p> <p>0</p> <p>0</p>	<p>IDENT</p> <p>TYPE</p> <p>RIN</p> <p>RIPAR</p> <p>2</p> <p>IIPAR</p>
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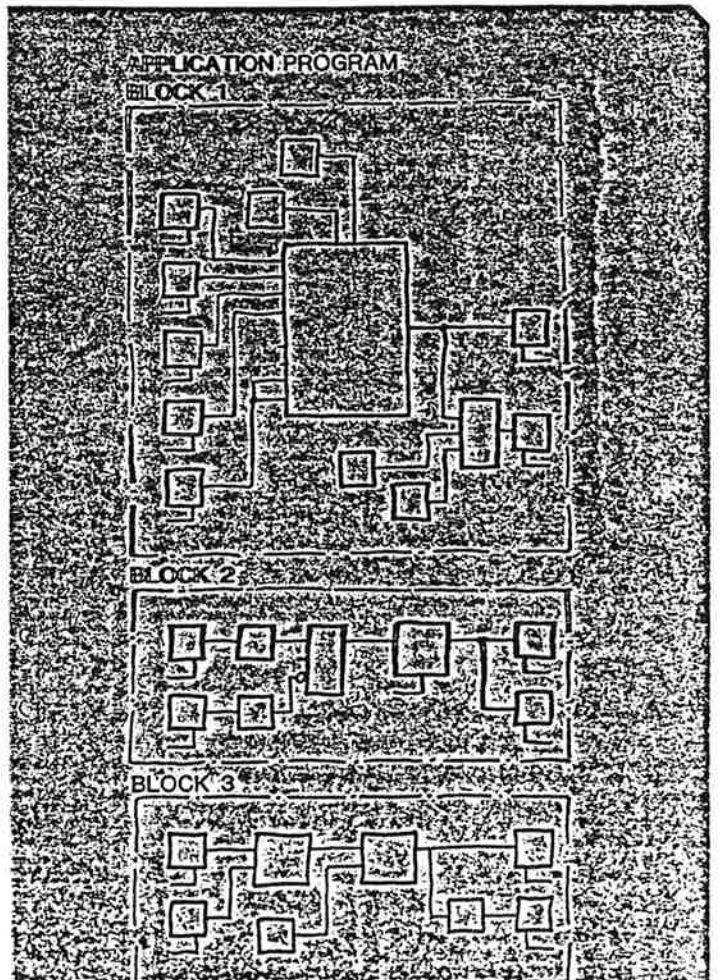
- ① Module identity is only defined in a block. I.E. 1 can be the first module even in block 2.
- ② Signals are general. I.E. 1 can only be used in this block.

DEFINING A BLOCK







<p>PGM</p> <p>① NEW</p> <p>1</p> <p>② TIME</p> <p>③ 4</p> <p>④ 1</p>	<p>IDENT</p> <p>TYPE</p> <p>PERIOD</p> <p>PRIORITY</p>
--	--

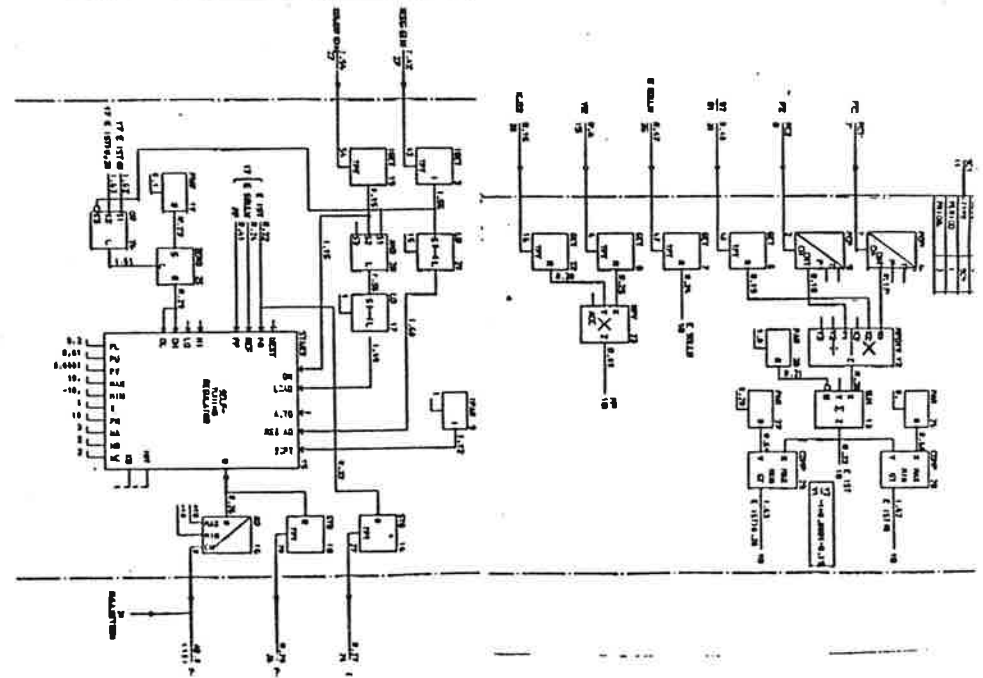
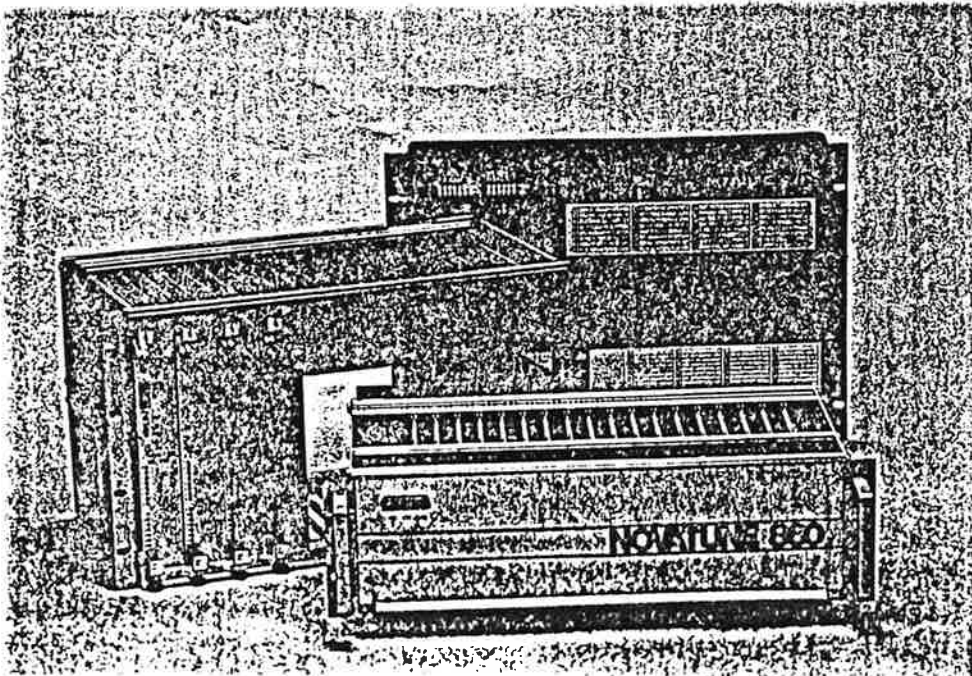
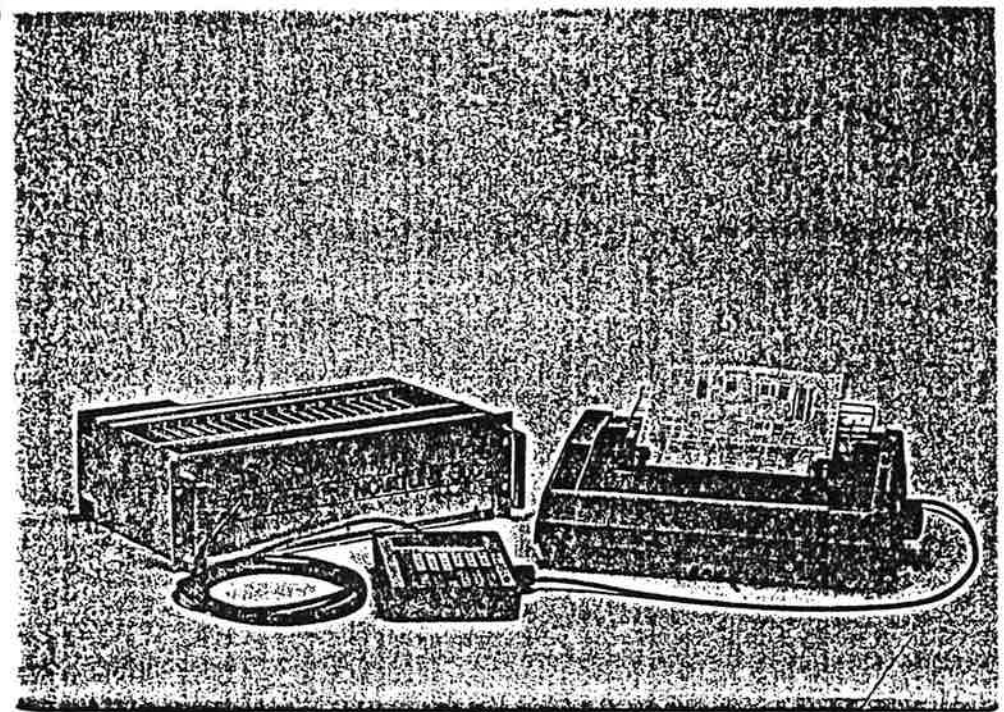
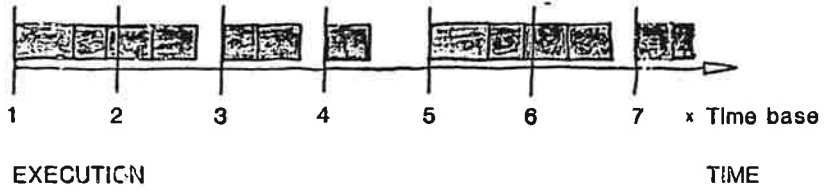
- ① NEW > New block is defined
- OLD > Already defined block is asked for
- DEL > Block is going to be deleted
- ② TIME > Time clock executed block
- SC1 > Soft clock 1 is executing the block (connected to sint module)
- ③ 4 > 4 base execution time

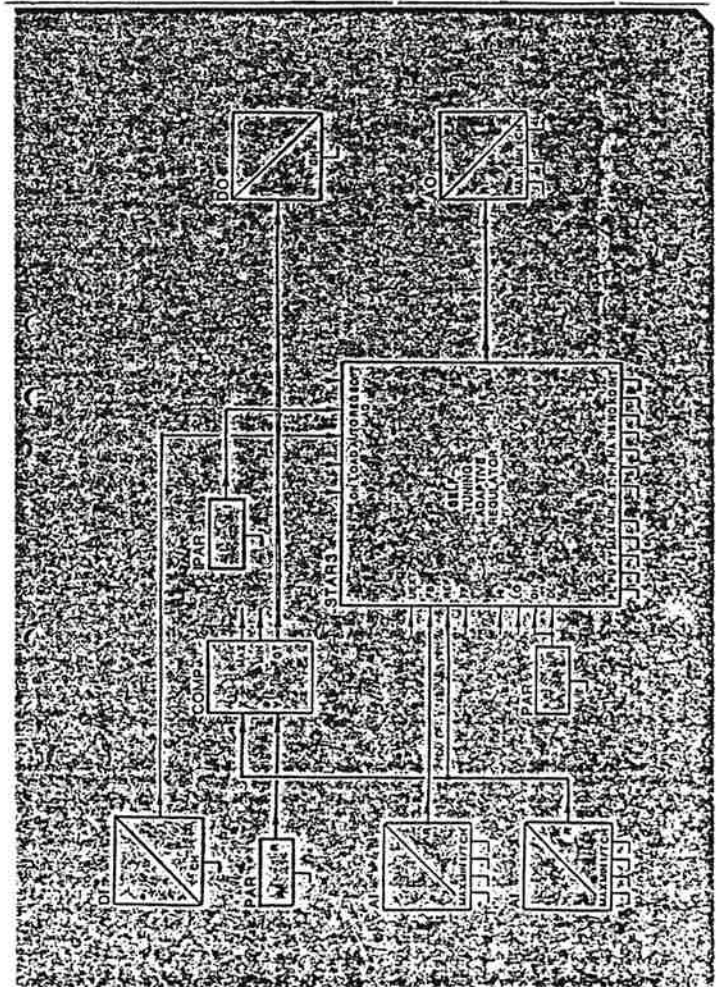
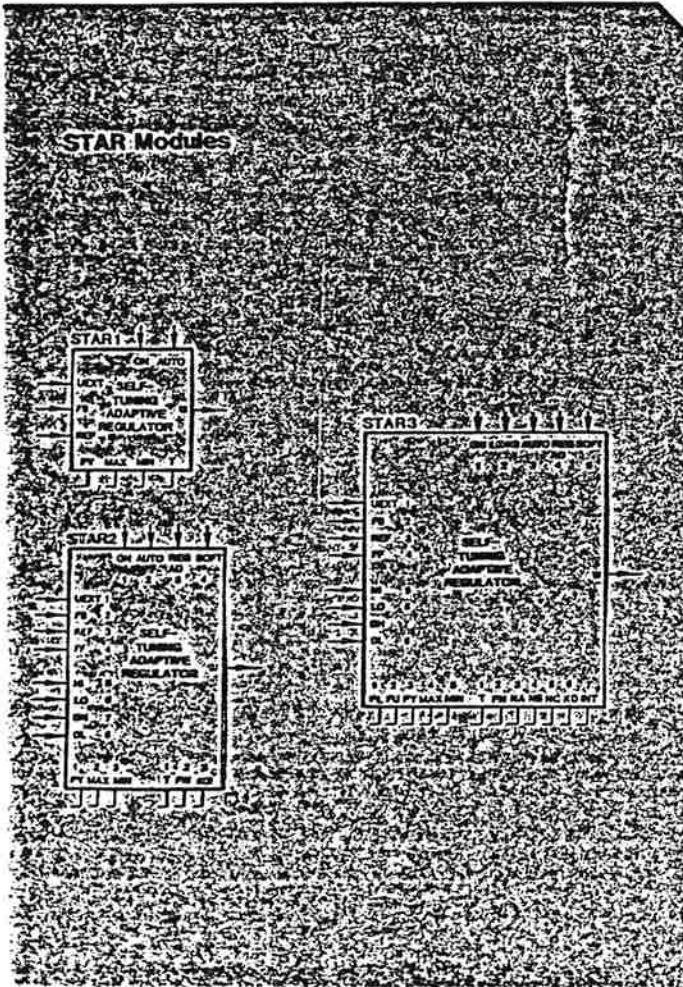
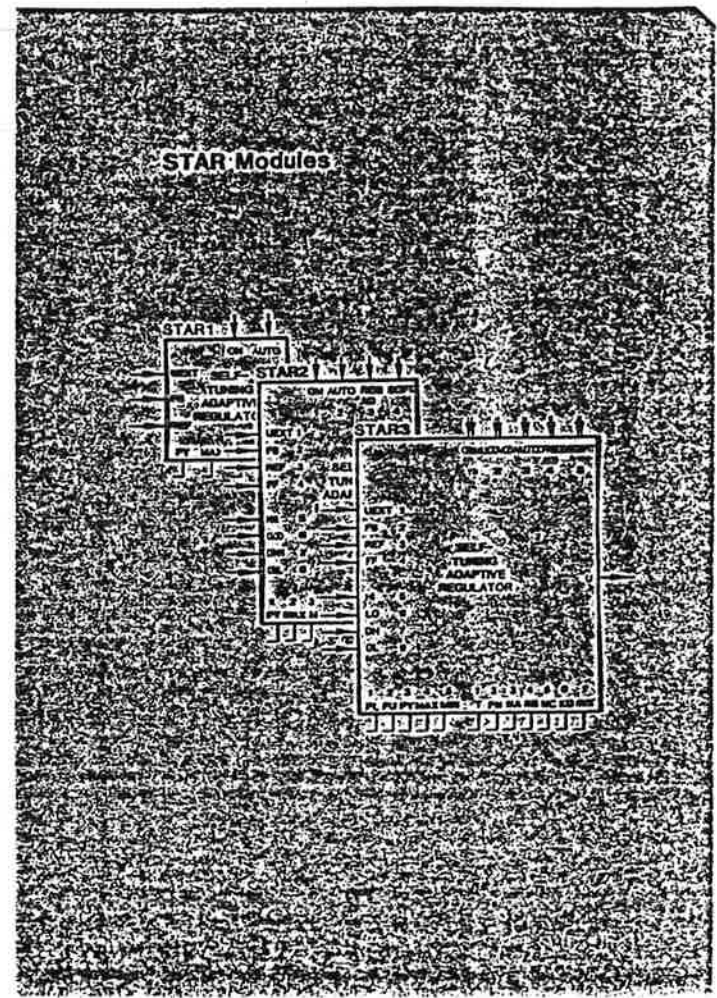
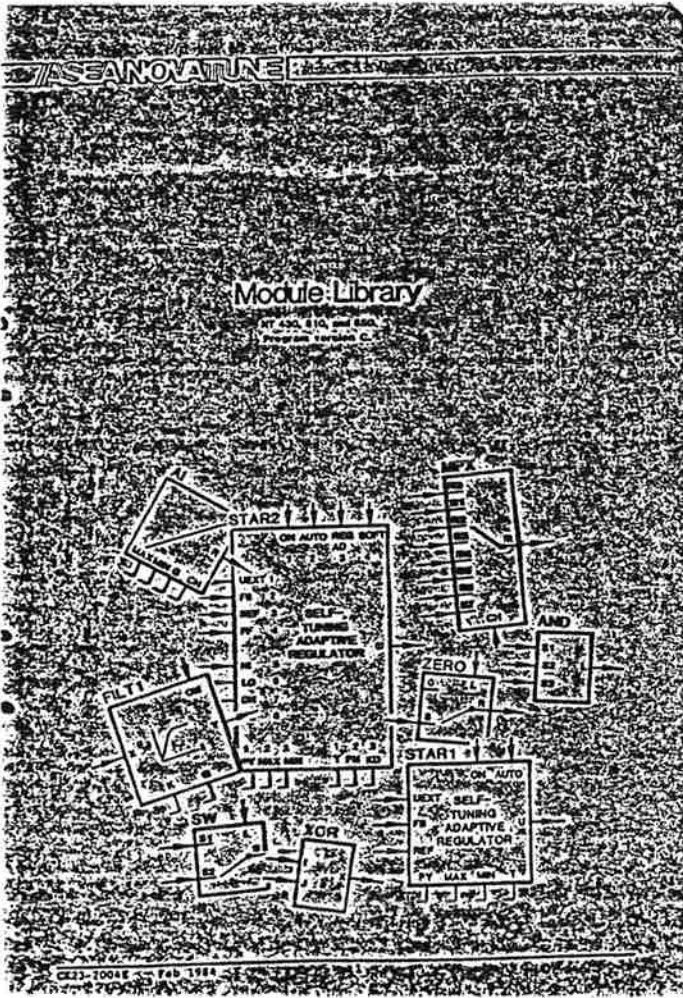


④ PRIORITY

EXAMPLE

BLOCK 1 PRIORITY 1 PERIOD 4 
 BLOCK 2 PRIORITY 2 PERIOD 2 
 BLOCK 3 PRIORITY 3 PERIOD 1 





Experiences with the ASEA-Novatune

Gunnar Bengtsson

**ASEA AB
Västerås, Sweden**

A number of feasibility studies for adaptive controls have been made at Asea during the past 10 years. The adaptive system Novatune was announced in 1982. A lot of experience has been gained during the past years use of it. About 70 systems are currently in operation. The systems cover a wide range of applications in paper mills, steel mills, boilers, waste water treatment, building automation, and looms. The installations of Novatunes is currently increasing rapidly. To do this it has been necessary to train a number of people in the commissioning of the system. A series of courses has been designed to give the proper background for customers and ASEA engineers.

There are several reasons why the Novatune system has got a good acceptance. In many process industries ordinary PID regulators are frequently badly tuned. As a result of this they are often switched to manual mode. There are a number of critical control loops where there are tangible economical benefits by reducing variations in quality variables. Experience indicates that reductions in standard deviations by a factor of two compared with a welltuned PID is quite common. The reason for this is that there frequently are time delays which the Novatune handles better than PID. The improvements in comparisons with poorly tuned PID are of course more favorable. Improvements in variances with an order of magnitude have been found in several cases when feedforward can be applied. Effective use of feedforward requires however good models which have to be updated. It is thus a good case for adaptive control.

Two applications are described in some detail, a cold rolling mill and a chemical reactor.

The rolling mill is a typical batch process there are roughly speaking three phases, startup, full speed operation and breaking. The Novatune was applied to the gauge control loop. The screw position was controlled using feedback from a gauge sensor after the rolls and by feedforward from a gauge sensor in front of the rolls. The main disturbances are variations in gauge and hardness. The time constants and the time delay varies with a factor of 25 over the operation range. The variations in the time delay are handled by having a speed sensor and by introducing length as the independent variable instead of time. The actual sampling period in the regulator will thus vary with speed from 40 ms at full speed to several seconds at slow speeds. The adaptive regulator performed significantly better than a conventional PID regulator with feedforward.

The first Novatune application was made in connection with temperature control in a chemical reactor. The temperature fluctuations were reduced by an order of magnitude mainly due to feedforward operation. The application is critical with

respect to safety and production. The possibility of storing parameters which will give a safe performance of the closed loop system and reinitializing the adaptation using these parameters was incorporated in the system. This application is described in more detail in [1].

The key problem areas that have been found have to do with nonlinearities like friction, dead zone and hysteresis. It may be a good idea to have more flexible ways of modeling these in the regulator. The unstable zeros which appear when the sampling rate is increased is another problem of practical importance.

Reference:

- [1] Bengtsson G, Egardt B (1984): Experiences with Self-Tuning Control in the Process Industry. Preprints of IFAC 9th World Congress, Budapest, Hungary, vol XI, 132-140.

Adaption to changing dynamics



Adaptive feedforward



Dead-Time



ASEA NOVATONE

are now running
in several plants

- Pulp Mills
- Pulp Driers
- Winder
- Chemical Reactor
- Skim Pass Mills
- Cold Rolling Mills
- Rotary Kilm Driers

REFERENCE LIST - 1984-03-10

STEEL/METALLURGIC

Krupp Bochum (Germany)	-	Skin Pals Mill
Grianges (Sweden)	-	Strip Tension
Sandvik (Sweden)	-	AGC
SSAB (Sweden)	-	Wouid Level Control
Falk (Italy)	-	Induction Furnace
Arvedi ()	-	- * -

PULP/PAPER

Marrum (Sweden)	-	Pulp Drying
Edst ()	-	Consistency Control
ERA (France)	-	Wetstetion Control
Byite (Sweden)	-	Roll Trimmer
Evarnsveden (Sweden)	-	- * -
St. Regis (USA)	-	- * -
Boise Caslade (Canada)	-	- * -
Bowater (Canada)	-	- * -
Albury (Australia)	-	- * -
Mandi (South Africa)	-	- * -
Hallsta (Sweden)	-	- * -
Follum (Norway)	-	- * -

FOOD

SEA (Sweden)	-	Beet Pulp Dryer
African Products (South Africa)	-	Maize Drying

CHEMICAL

Berol (Sweden)	-	Chemical Reactor
Telleburl (Sweden)	-	Rubber
Cements (Sweden)	-	Cement Klinker
Berol (Sweden)	-	Total Instrumentation

Boliden (Sweden)	-	Total Instrumentation
Kamared (Sweden)	-	- * -

BOILER

Roskilde (Denmark)	-	Boiler Control
Manusso (Sweden)	-	Total Instrumentation

WATER

Kappala (Sweden)	-	Waste Water Control
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BUILDING AUTOMATION

Huddinge Hospital (Sweden)	-	Total Instrumentation
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OPEN

Alrnuit (Global)	-	Loam Control
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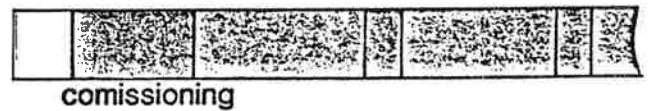
ASEA NOVATUNE
Improves the performance
of your control system.

- increased production capacity
- decreased production costs
- improved quality
- reduced installation costs
- reduced maintenance costs

Reduced Installation Costs



Conventional controllers:



ÅSEA NOVATUNE controllers:



 disturbed production


 undisturbed production

Figure 7 Self-adaptive regulator with offset distance on the entry gauge measurement. The regulator reduces its feedback

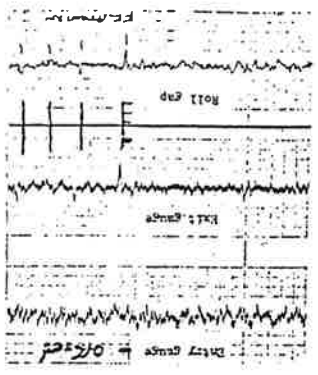


Figure 6 The first pass with the self-adaptive regulator controlling a strip with large deviation in the entry gauge. The tolerance limit (in terms of deviation) after the ninth pass are marked out

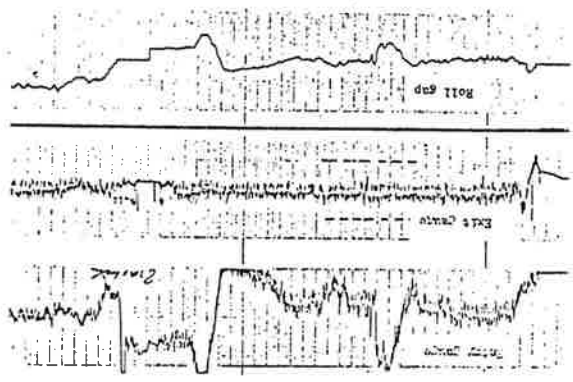


Figure 8 Self-adaptive regulator with severe mechanical interaction. Discrepances in the entry gauge due to

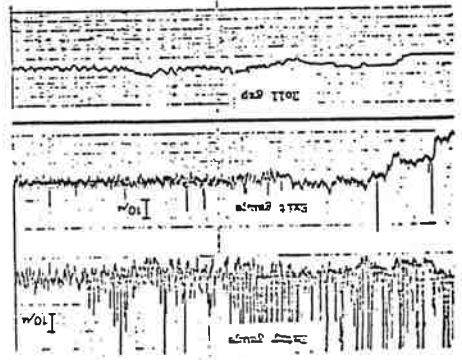


Figure 5 Milling with a self-adaptive regulator

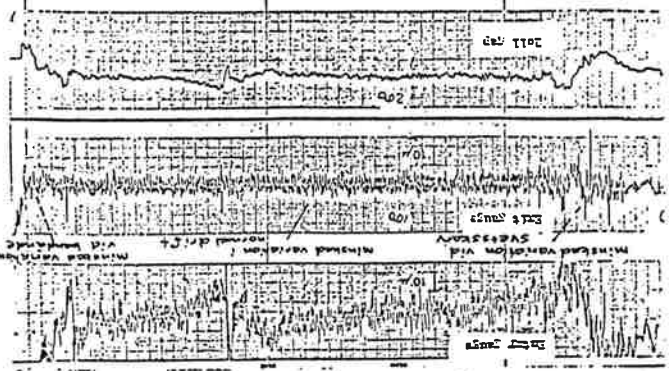
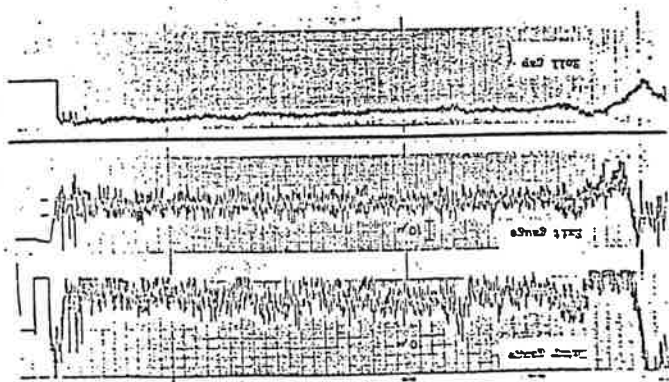


Figure 6 Milling with a well-tuned conventional regulator



ASZA

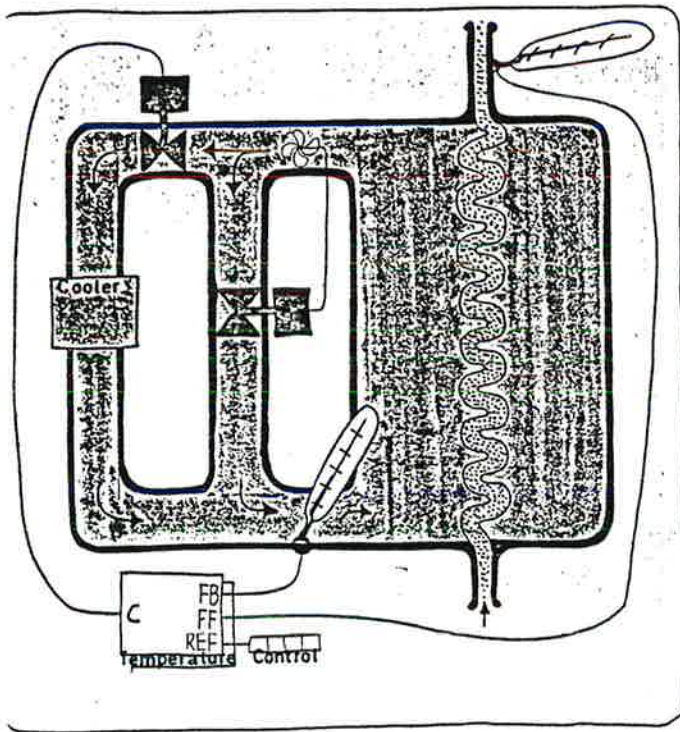
PROBLEMS
Speed
Hardness
Deformations

REF
FF
FB

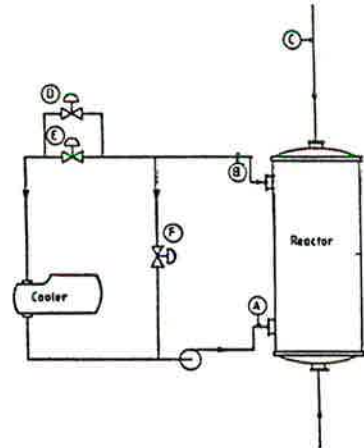
NOMATURE

GOLD ROLLING MILL

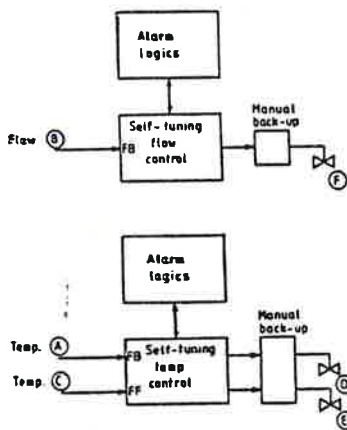
Chemical Reactor



Reactor Design

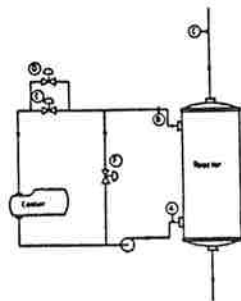
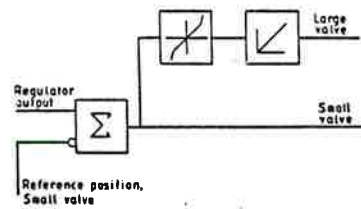


Reactor Control System Structure



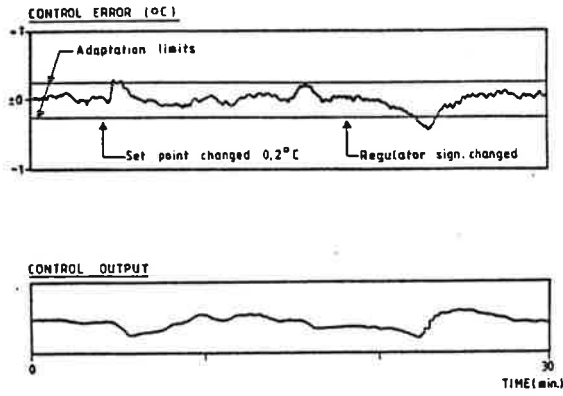
Reactor Control

Combining parallel valves

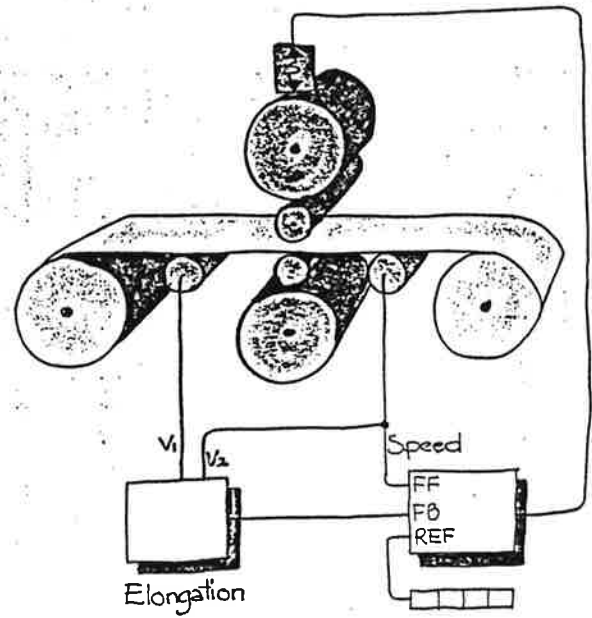


Reactor Control

Performance

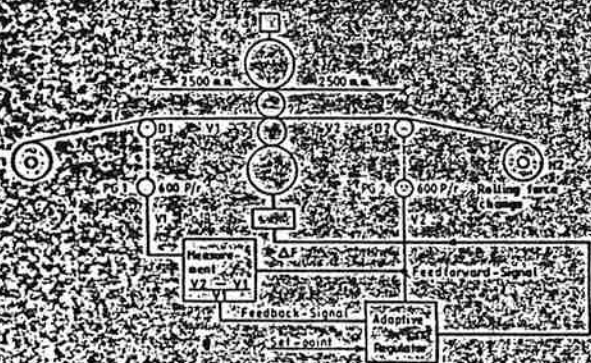


Skin Pass Mill



Elongation Control

Design Principle

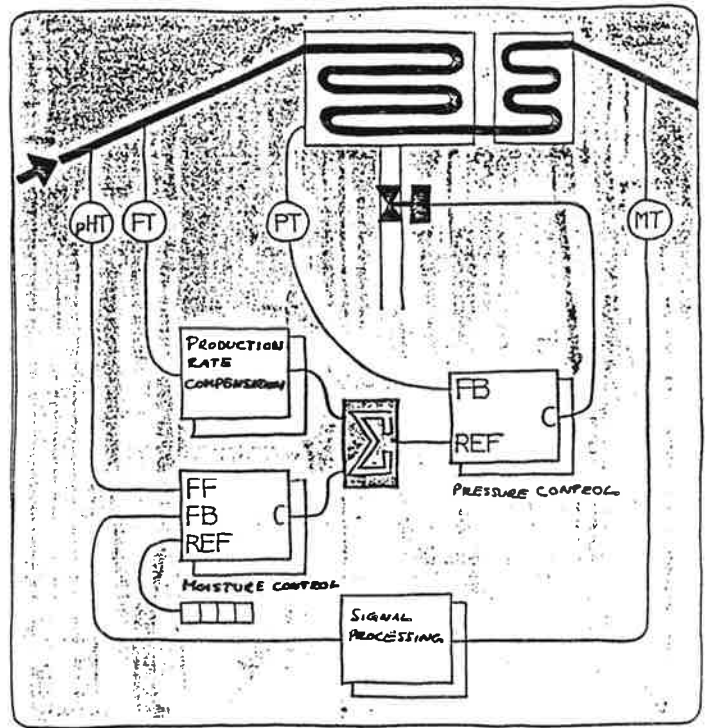


Elongation Control

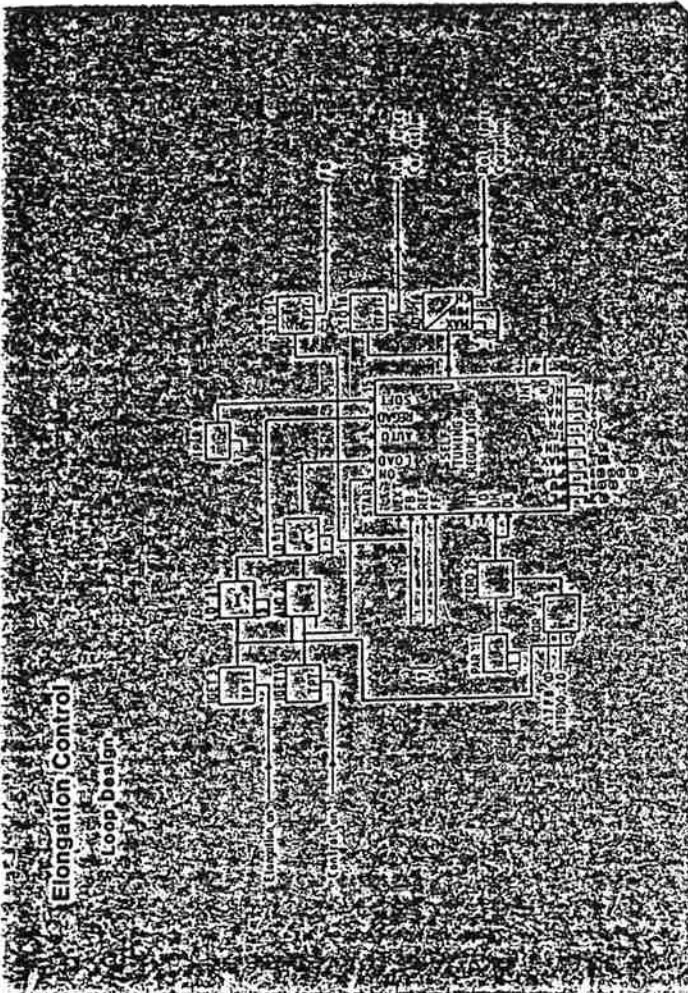
Performance



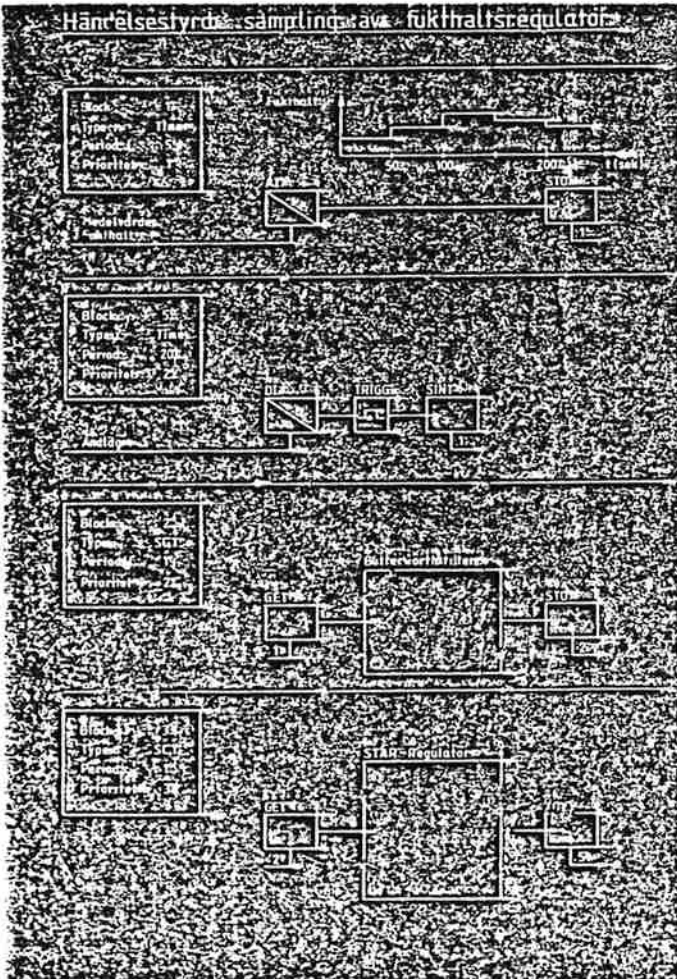
PULP DRIER



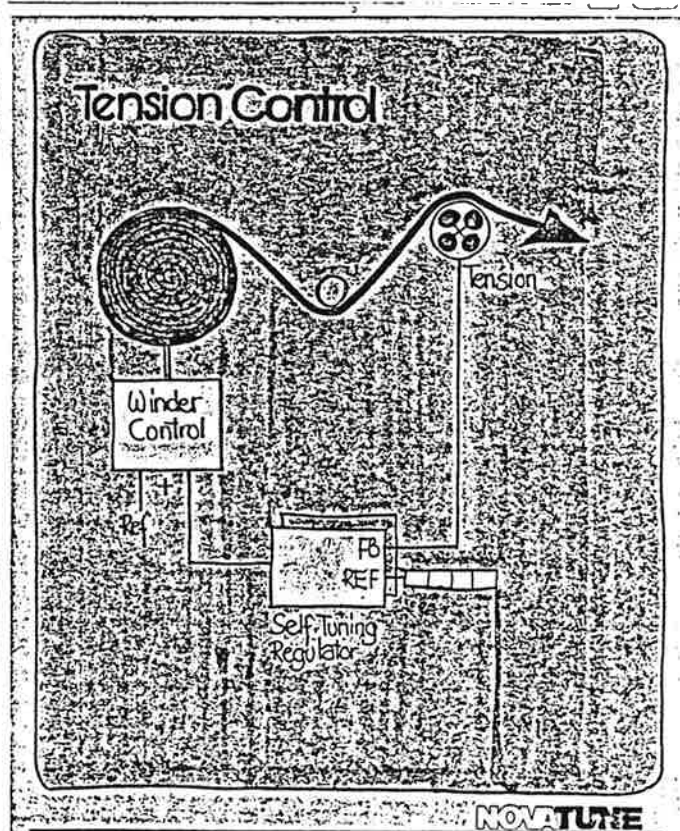
Elongation Control Loop Design



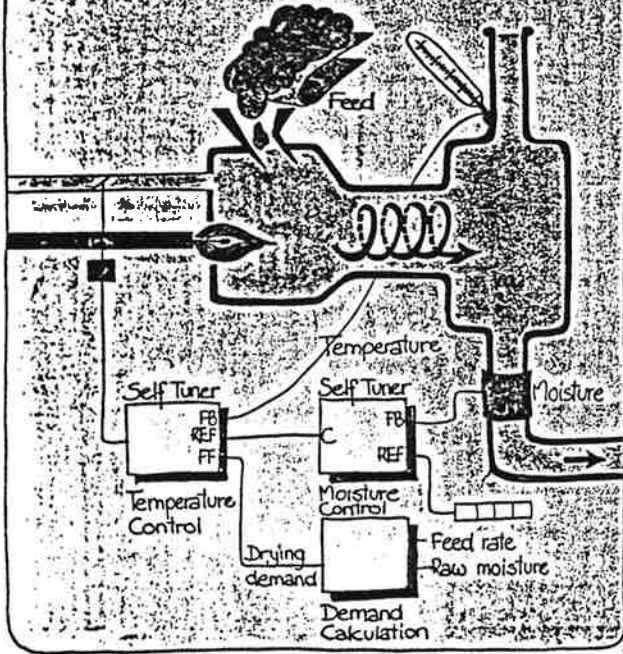
Hänfelsesstyrning, samlings- och fukthaltsregulator



Tension Control



Rotary Drum Drier



NOVATURE
ASEA

Automatic tuning of simple regulators

K.J. Åström and T. Hägglund

Department of Automatic Control
Lund Institute of Technology
Lund, Sweden

Abstract.

Procedures for automatic tuning of regulators of the PID type are described. The methods are based on a simple identification method which gives critical points on the Nyquist curve of the open loop transfer function. The key idea is a scheme which provides automatic excitation of the process which is nearly optimal for estimating the desired process characteristics. The methods proposed are primarily intended to tune simple regulators of the PI(D) type. In such applications they will of course inherit the limitations of the PI(D) algorithms. They will not work well for problems where more complicated regulators are required. The proposed algorithms may be used in several different ways. They may be incorporated in single loop controllers to provide an option for automatic tuning. They may also be used to provide a solution to the long-standing problem of safe initialization of more complicated adaptive or self-tuning schemes. In contrast to other methods based on self-tuning control, they do not require a priori information about time scales.

References

- Åström, K. J. and T. Hägglund. Automatic Tuning of Simple Regulators with Specifications on Phase and Amplitude Margins. Proceedings IFAC Workshop on Adaptive Systems in Control and Signal Processing. San Francisco 1983.
- Åström, K. J. and T. Hägglund. Automatic Tuning of Simple Regulators with Specifications on Phase and Amplitude Margins. *Automatica* **20**, 645-651, 1984.
- Åström, K. J. and T. Hägglund. Automatic Tuning of Simple Regulators. Proceedings IFAC 9th World Congress, Budapest, Hungary. 1984.

AUTOMATIC TUNING OF SIMPLE REGULATORS

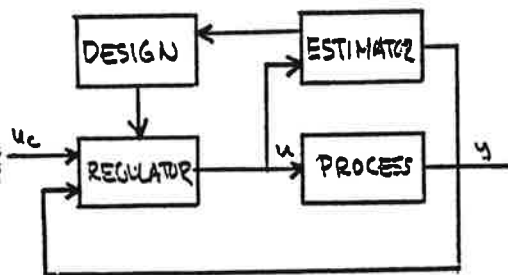
K.J. Åström
LUND - SWEDEN

1. INTRODUCTION
2. THE BASIC IDEA
3. CONDITIONS FOR OSCILLATION
4. REGULATOR DESIGN
5. PRACTICAL ISSUES
6. EXPERIMENTS
7. CONCLUSIONS

INTRODUCTION

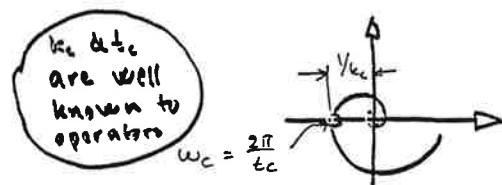
- ✿ BACKGROUND
 - ADAPTIVE CONTROL RESEARCH
 - ROBUSTNESS
 - UNMODELED DYNAMICS
 - PRIOR INFORMATION
 - REACTIONS FROM INDUSTRY
- ✿ ADAPTATION VS TUNING
- ✿ THE PID STRUCTURE
- ✿ STR & MRAS APPROACHES
 - LS+MU
 - LS+PP
 - ELS+LQG
- ✿ A NEW APPROACH

SELF-TUNING CONTROL



THE BASIC IDEA

- ✿ DESCRIBE THE PROCESS IN TERMS OF CRITICAL GAIN k_c AND CRITICAL PERIOD t_c .



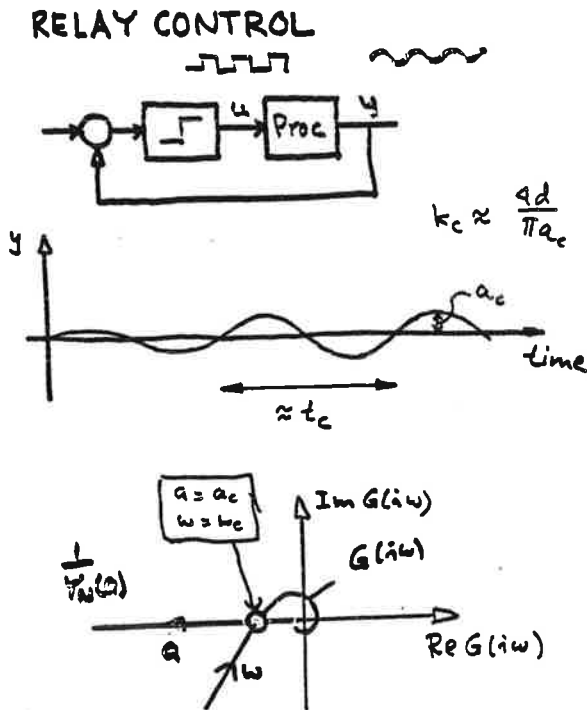
- ✿ FIND METHODS FOR DETERMINING k_c AND t_c .

- ✿ APPLY DESIGN METHOD BASED ON k_c AND t_c

EX: ZIEGLER NICHOLS

$$k = \frac{k_c}{2}, T_i = \frac{t_c}{2}, T_d = \frac{t_c}{8}$$

DETERMINATION OF k_c & t_c



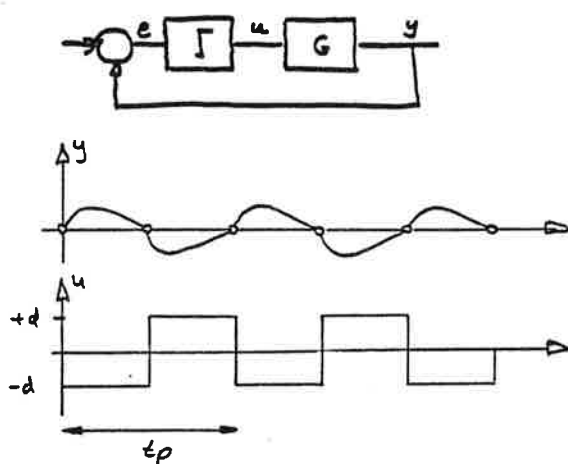
4

PROPERTIES

- ✦ AUTOMATIC GENERATION OF NEAR OPTIMAL INPUT
- ✦ GOOD POSSIBILITIES FOR CONTROLLING THE OUTPUT AMPLITUDE DURING THE EXPERIMENT
- ✦ SAFE PROCEDURE FOR STABLE SYSTEMS

5

CRITERIA FOR OSCILLATIONS



SAMPLE AT $t_p/2$

$$Z(u) = \frac{1}{z+1}$$

THEOREM: $H(-1) = 0$

APPROXIMATION

$$k_c \approx \frac{4d}{\pi a}$$

6

THE DESCRIBING FCN APPROX.

$$H(e^{sh}) = \sum_{n=-\infty}^{\infty} \frac{1}{h(s+i\omega_n)} (1 - e^{-h(s+i\omega_n)}) G(s+i\omega_n)$$

$$\omega_n = \frac{2\pi}{h}, \quad sh = i\pi \text{ gives}$$

$$H(-1) = \sum_{n=-\infty}^{\infty} \frac{2}{i(\pi+2n\pi)} G\left(i \frac{\pi+2n\pi}{h}\right)$$

$$= \sum_{n=0}^{\infty} \frac{4}{\pi(1+2n)} \text{Im} \left\{ G\left(\frac{\pi+2n\pi}{h} i\right) \right\}$$

$$\approx \frac{4}{\pi} \text{Im} \left\{ G\left(i \frac{\pi}{h}\right) \right\}$$

5

EXAMPLE 1

$$G(s) = \frac{1}{s} e^{-sT}$$

$$H(z) = \frac{z^T + (h-T)}{z-1}$$

$$H(-1) = 0 \Rightarrow h = 2T$$

$$\text{Period } T_p = 2h = \underline{\underline{4T}}$$

$$\arg G(i\omega_c) = -\frac{\pi}{2} - \omega_c T = -\pi$$

$$\Rightarrow \omega_c = \frac{\pi}{2T} \quad T_p = \frac{2\pi}{\omega_c} = \underline{\underline{4T}}$$

EXAMPLE 2

$$G(s) = \frac{1}{s(s+1)(s+a)}$$

$$\arg G(i\omega_c) = 0 \Rightarrow \omega_c = \sqrt{a}$$

$$H(-1) = -\frac{h}{2a} + \frac{1}{a-1} \left[\frac{1-e^{-h}}{1+e^{-h}} - \frac{1}{a^2} \frac{1-e^{-ah}}{1+e^{-ah}} \right]$$

$$H(-1) = 0 \Rightarrow h \approx \frac{2\sqrt{3}}{\sqrt{a}}$$

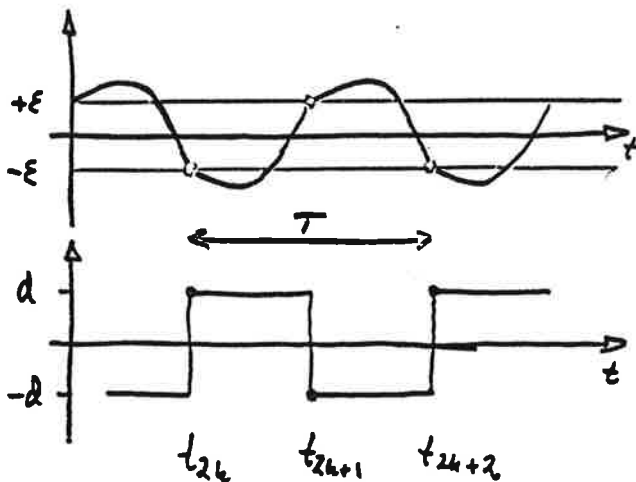
$$T_p = \frac{2\pi}{\omega_c} = \frac{6.28}{\sqrt{a}} \quad a_{\text{max}}$$

$$T_p \approx \frac{4\sqrt{3}}{\sqrt{a}} = \frac{6.92}{\sqrt{a}}$$

6C

6D

CONDITIONS FOR PERIODIC SOLUTIONS



$$\mathcal{Z}\{u\} = \frac{d}{z+1} \quad \mathcal{Z}\{y\} = -\frac{\epsilon}{z+1}$$

$$H\left(\frac{z}{2}, -1\right) = -\frac{\epsilon}{d}$$

EXAMPLE:

$$G(s) = \frac{b}{s+a} \quad a, b > 0$$

$$H(\tau, z) = \frac{b(1-e^{-a\tau})}{z-e^{-a\tau}}$$

$$H(\tau, -1) = -\frac{b(1-e^{-a\tau})}{1+e^{-a\tau}} = -\frac{\epsilon}{d}$$

$$T = 2\tau = -\frac{2}{a} \ln \frac{bd-\epsilon}{bd+\epsilon} \approx \frac{4\epsilon}{abd}$$

10

35

EXAMPLE

$$\frac{dx}{dt} = \begin{bmatrix} 0 & \omega \\ -\omega & 0 \end{bmatrix} x + \begin{bmatrix} 0 \\ \epsilon \end{bmatrix} u$$

$$y = [1 \ 0] x$$

$$H(\tau, z) = \frac{z - 2\cos\omega\tau + 1}{z^2 - 2z\cos\omega\tau + 1}$$

$$H(\tau, -1) = -\frac{\cos\omega\tau}{1 + \cos\omega\tau} = -\frac{\epsilon}{d}$$

$$T = 2\tau = \frac{2}{\omega} \arccos \frac{\epsilon}{d - \epsilon}$$

$$\epsilon = 0 \Rightarrow T = \frac{2}{\omega} \cdot \frac{\pi}{2} = \frac{\pi}{\omega}$$

66

HOW TO DETERMINE t_c & a

ZERO CROSSINGS &
PEAK DETECTION

ZERO CROSSINGS &
CORRELATION

LEAST SQUARES

$$\sum [y(t) - by(t-h) + y(t-2h)]^2 \min$$

$$b = 2 \cos(h/t_c) \cdot 2\pi$$

$$\sum [y(t) - a_1 \sin \omega t - a_2 \cos \omega t]^2$$

$$\omega = 2\pi/t_c$$

7

EXAMPLE:

$$G(s) = \frac{1}{s^2}$$

$$H(\tau, s) = \frac{\tau^2}{2} \frac{z+1}{(z-1)^2}$$

$$H(\tau, -1) = 0$$

✿ NO PERIODIC SOLUTION WITH
HYSTERESIS

✿ PERIODIC SOLUTIONS WITH
ARBITRARY PERIOD WITH
IDEAL RELAY

67

ZIEGLER - NICHOLS RULE 1

CRITICAL GAIN K_c
PERIOD T_c

C

C

	K	T _I	T _D
P	0.5K _c		
PI	0.4K _c	0.8T _c	
PID	0.6K _c	0.5T _c	0.12T _c

C

C

MODIFIED RULES:

BETTER DAMPING!

8

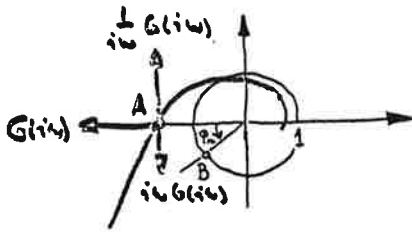
CONTROL DESIGN

✦ AMPLITUDE MARGIN DESIGN

$$G_R(s) = k \left[1 + \frac{1}{sT_i} + sT_d \right]$$
$$= k \left[1 + \frac{1}{sT_i} (1 + s^2 T_i T_d) \right]$$

$$k = \frac{k_c}{A_m}, \quad T_d = \frac{1}{\omega_c^2 T_i}, \quad T_i \text{ arbitrary}$$

✦ PHASE MARGIN DESIGN



PICK k , T_i AND T_d TO MOVE A TO B

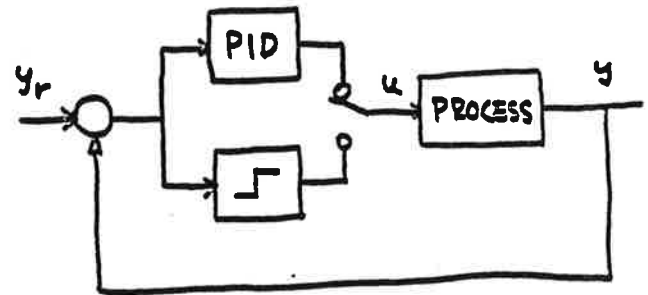
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PRACTICAL ISSUES

- ✦ MEASUREMENT NOISE
- ✦ ADJUSTMENT OF AMPLITUDE OF OUTPUT
- ✦ SATURATION OF ACTUATORS
- ✦ WHAT PRIOR INFORMATION IS NEEDED?
- ✦ HYSTERESIS

11

REGULATOR STRUCTURE



11

EXPERIMENTS

- ✦ GOALS
 - WHEN & HOW WILL IT WORK?
 - USER REACTION
- ✦ MEANS
 - LSI 11/03, Apple II,
 - Intel 8086
 - Analog computer simulation
 - Laboratory processes
 - Industry
 - Flow
 - Temperature
 - Level
 - Composition

✦ RESULTS

11

PI - AUTO-TUNER

$$G(s) = \frac{1}{(1+0.25s)^4}$$

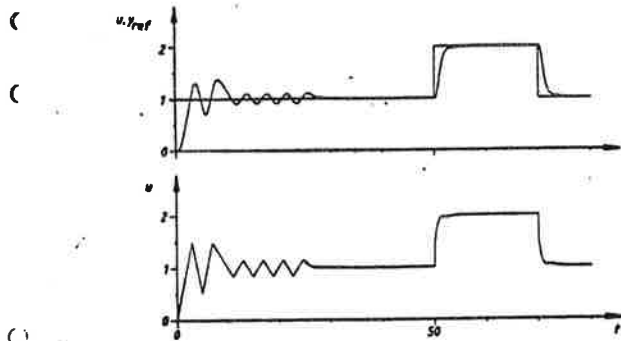


Figure 9. - Simulation of a PI auto-tuner applied to a process with the transfer function $G(s) = 1/(1+0.25s)^4$.

VARIATIONS IN PROCESS GAIN

$$G(s) = \frac{k}{(1+0.25s)^4}$$

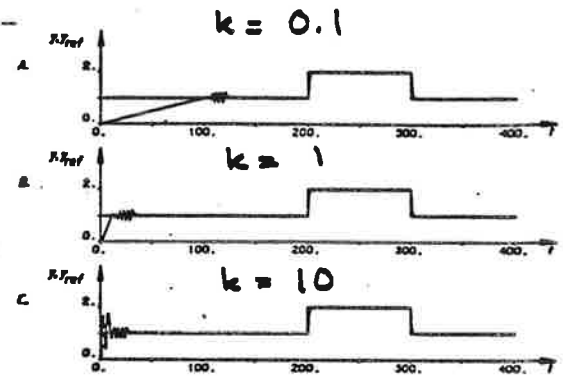


Figure 10. - Simulation of a PI auto-tuner applied to a process with the transfer function $G(s) = k/(1+0.25s)^4$. The process gain k is 0.1, 1 and 10 in A, B, and C respectively.

13

VARIATIONS IN PROCESS TIME CONSTANTS

$$G(s) = \frac{1}{(1+sT)^4}$$

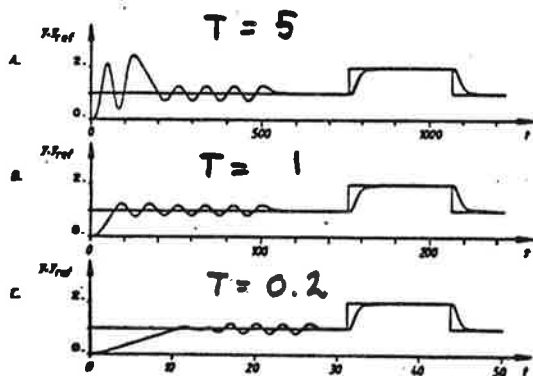


Figure 11. - Simulation of a PI auto-tuner applied to a process with the transfer function $G(s) = 1/(1+sT)^4$. The process time constant T is 5, 1 and 0.2 in A, B, and C respectively.

EFFECTS OF NOISE

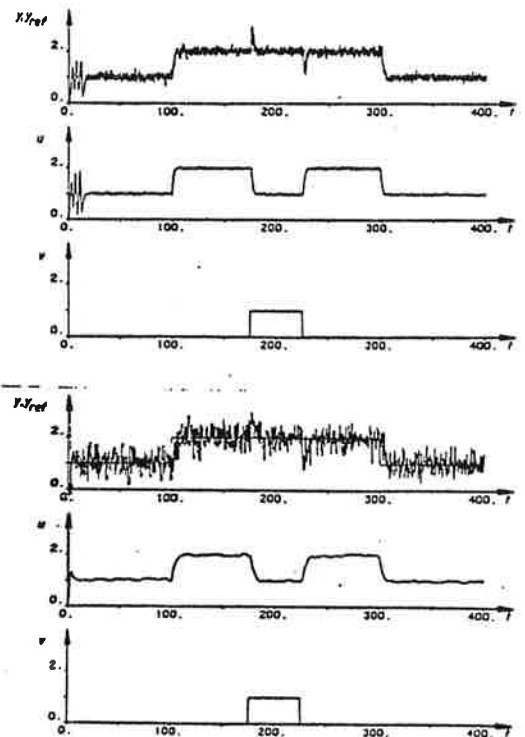
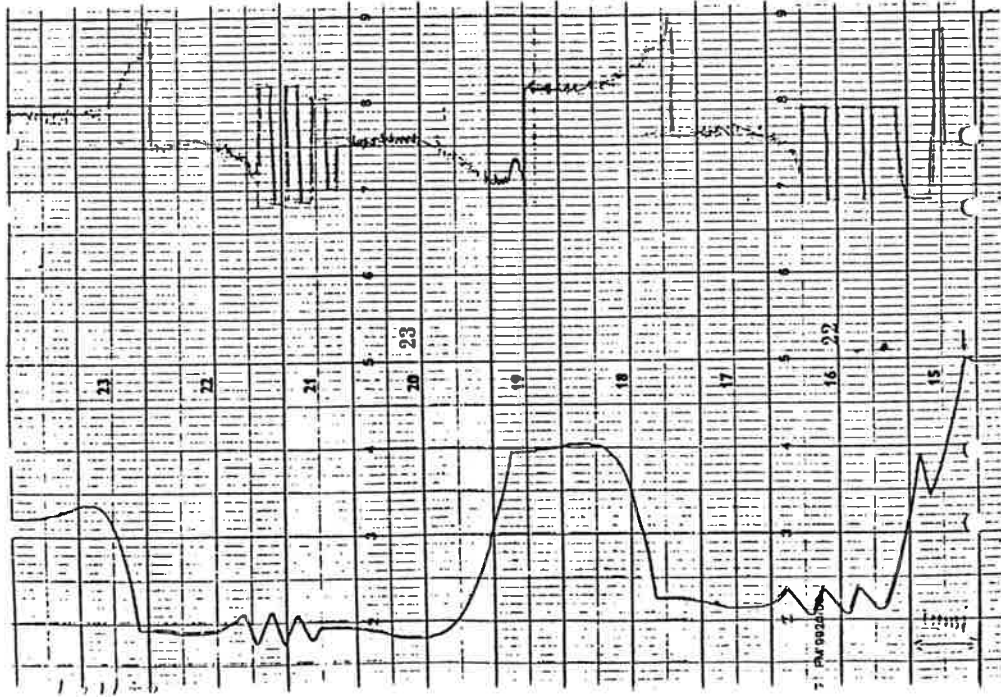


Figure 12. - Simulation of the system with variable noise level. The standard deviation of the measurement noise is 0.1 and 0.3 in A and B respectively.

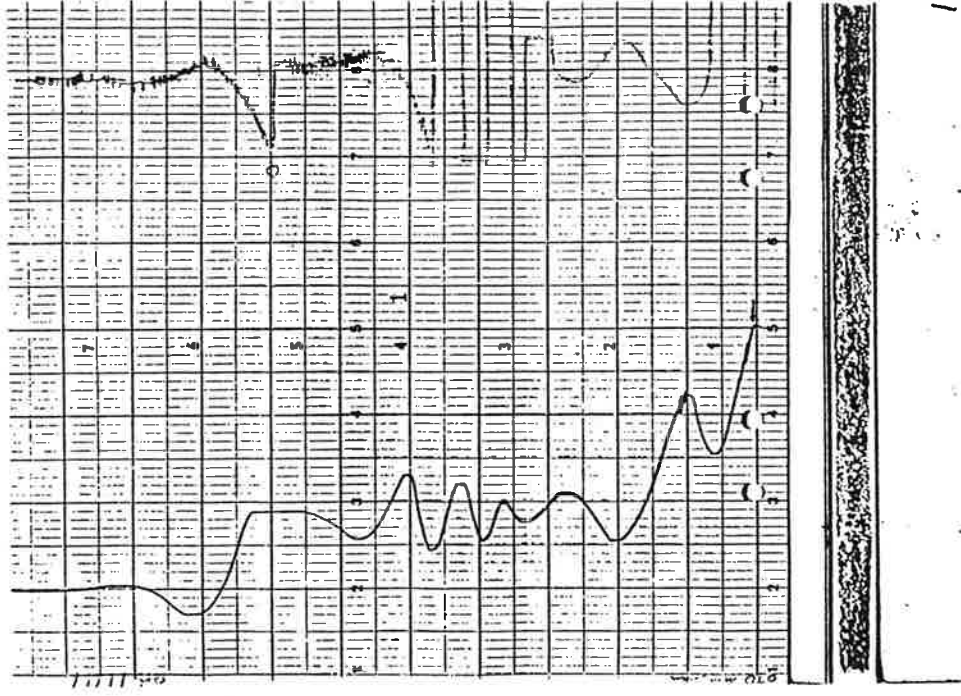
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16



CONCLUSIONS

- ⊗ A SIMPLE IDEA
- ⊗ SEEM TO WORK WELL
- ⊗ ROBUST
- ⊗ IMPLICATIONS FOR MORE GENERAL ADAPTIVE SYSTEMS
- ⊗ GETS YOU IN THE BALL PARK
- ⊗ RESEARCH ISSUES



AUTOMATIC TUNING OF

SIMPLE REGULATORS

K. J. Åström
LUND - SWEDEN

1. INTRODUCTION
2. THE BASIC IDEA
3. AMPLITUDE-MARGIN DESIGN
4. PHASE-MARGIN DESIGN
5. EXPERIMENTS
6. CONCLUSIONS

The NAF - Autotuner

Lars Bååth

NAF Controls AB
Stockholm

Among other products for instrumentation and control NAF Control AB manufactures the Control and Information system NAF-Unic. As a special version of the software of the systems NAF-Unic S SDM-10 and NAF-Unic S SDM-20 the NAF-Autotuner can be obtained.

The NAF-Unic S SDM-20 system consists of one central unit, one or two color screens and function keyboards, one alphanumeric keyboard for program development, one printer and one tape recorder. The system has the capacity of handling 30 analog and 30 digital inputs, 16 digital and 16 analog outputs and up to 45 PID-controllers. With 45 PID-loops the system loop time is 250 ms. The expanded version can handle 240 inputs and 128 output signals.

The system can be programmed with function modules such as PID-controllers, a deadtime controller, limiters, alarm modules, adders, multipliers, logical modules and so on. In total there are about 30 different modules. The software of the system can contain 220 function modules. The system also contains a PLC-system which is integrated with the analog control system. It is possible to program the system on line.

A major goal in the design of this system has been high system security. Several measures have been taken to achieve this, for example

- All PC-boards are duplicated
- Checksum calculation of configuration in RAM and code in EPROM
- A triple serial bus for internal communication is used
- Self diagnostic routines for the inputs and outputs

The NAF-Unic S SDM-20 and SDM-10 systems also have incorporated an autotuner to help the operator in tuning the PID- controllers. A major design effort has been to simplify operation of the autotuner. All 45 PID regulator loops can use the Autotuner. The Autotuner has been developed by NAF-Controls in collaboration with Karl Johan Åström and Tore Hägglund at the Department of Automatic Control at Lund Institute of Technology. The principle of the Autotuner is to replace the normal PID-controller by a relay controller. A system with a relay controller starts to oscillate. If the period and amplitude of this oscillation are measured, the critical gain (k_c) and the critical period time (T_c) can be computed, because the describing function of the relay is known. When k_c and T_c are known a PID-controller can easily be designed.

The use of NAF-autotuner is very simple. Tuning can be started on operator command or by a digital signal. When no previous tuning is done, the operator must bring the process up to desired reference value in MAN-mode and start tuning.

If there is a controller that already has been tuned, just start tuning. Tuning can be interrupted by setting the controller in MAN- or AUTO-mode. When tuning is ready, the controller is automatically set in AUTO-mode with new PID-parameters.

SDM-20. System Features and Capacity

8

- handles both analogue and digital signals
 - 60 input signals – analogue or digital
 - 32 output signals – analogue or digital
- for up to 45 control loops
- supervision of up to 8 external single loop controllers
- optional signal can be connected to a pen-recorder for long-term trends
- system loop time, 250 ms
- software with 220 function modules
- advanced PLC-system for interlocking and sequence control
- expandable to 240 input and 120 output signals
- Very high system security
- PLC-system is integrated with analogue control-system by means of 256 digital signals used for communication
- On line programming

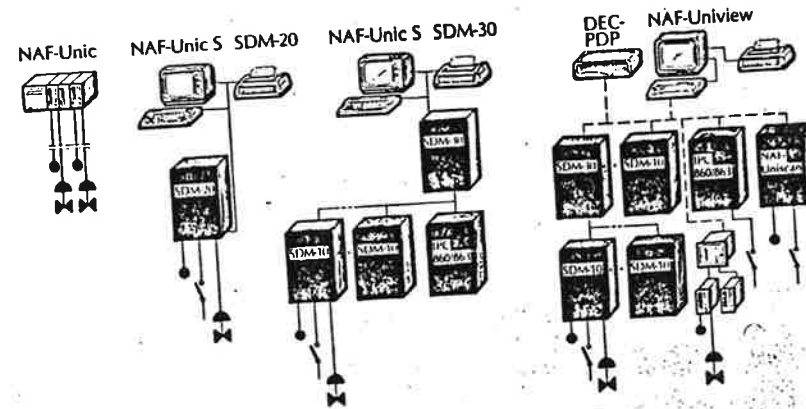


NAF-Unic S on the spot computing

SDM-20-13 o GB

NAF Control and Information System

4

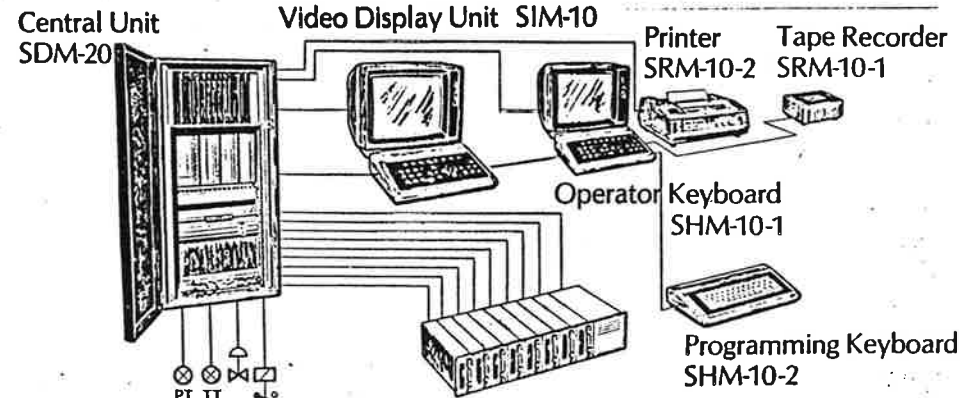


NAF-Unic S on the spot computing

SDM-1 o GB

NAF-Unic S SDM-20 with options

7



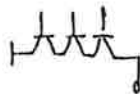
NAF-Unic S on the spot computing

SDM-20-11 o GB

Major Design Goal

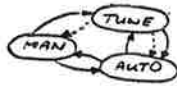
System Security

- Serial Bus (triplered)
- Backup on all PCB:s
- PCB:s with continuous selfcheck function and watch-dog
- All outputs overload protected
- 24V DC supply with battery backup available
- Alarm on PCB:s, VDU and printer on error
- Checksum calculation of configuration in RAM and code in EPROM



NAF - AUTOTUNER in Unic-5

- Major design effort : Simplify operation of AUTOTUNER
- Unic-5 incorporates 45 PID controllers with AUTOTUNER



OPERATION

- Tuning can be started on operator-command or by a digital signal
- Start of tuning is only permitted if control error is sufficiently small

First time tuning : Slowly bring up the process up to desired reference value, in MAN-mode, and start Tuning.

Previous tuning : Start Tuning

- Tuning ready \Rightarrow Controller automatically set in AUTO with new PID-parameters

- Tuning can be interrupted by setting controller in MAN- or AUTO-mode

TUNE \rightarrow MAN \Rightarrow Control signal returns to its value prior to start of tuning

TUNE \rightarrow AUTO \Rightarrow Controller returns to AUTO-mode with old PID-parameters

NAF - AUTOTUNER NAF-Unic 5

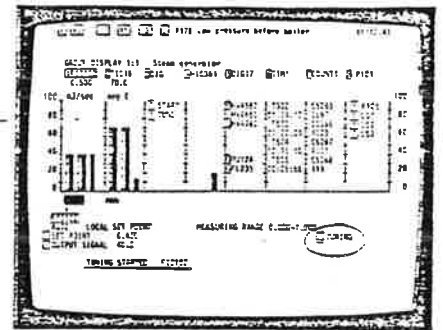


Fig. 2 Group display with controller selected for tuning

No.	Parameter	Note	Value	Unit
1	Function block			PID
2	Circuit reference		
3	Input signal	AI 1	
4	Measuring range, high		
5	Measuring range, low		
6	P gain		0.100000	
7	I ₁ integration time		1000.0	
8	I ₂ integration time		0.000000	
9	D differentiation time		0.250000	
10	Output signal	AO		
11	Remote set point	AI 2		
12	Start local set point			
13	Set point limiting, high			
14	Set point limiting, low			
15	Store current set points			
16	DI blocking, 1 sector	DI 1		
17	DI blocking, controller	DI 2		
18	Output signal, controller blocking (S)			
19	Output signal, zero deviation (S)			
20	DC remote/local	DC 1		
21	DC auto/manual	DC 2		
22	Control/reverse (L/R)			
23	Start remote/local			
24	Start auto/manual			
25	Start output signal (S)			
26	Control blocking (MAN)			
27	Start auto/manual	DC 3		
28	Start remote/local	DC 4		

History and issues in adaptive flight control

Gunter Stein

Honeywell Systems Research Center
Minneapolis, USA

This paper attempts to answer three questions: Why are there no adaptive flight control systems in modern aircrafts? Why has adaptive theory been irrelevant to all attempted designs? What is happening to change this around?

Inspection of aircraft physics reveals that the linearized aircraft dynamics depend typically on three parameters, dynamic pressure, Mach number and angle of attack. If these parameters are known it is straight forward to design flight control systems which accomplishes desired goals. Two competing alternative designs are discussed: the adaptive schemes and systems based on air data scheduling. The history is described from the point of view of competition between these schemes. An important aspect is that the control systems are flight critical in new aircraft. This means that rigid safety measures are required. The space shuttle is another example. It requires four identical control channels plus a fifth backup channel. These channels are not adaptive. The adaptive systems described include the ones used in the X15, F-111 and the F-8.

The tradeoff between performance and stability are reviewed in order to discuss quantitatively the influence of plant uncertainty on feedback design. The plant uncertainties are separated into structured uncertainty which corresponds to the rigid body dynamics and unstructured uncertainties which correspond to flexure modes. Adaptive control can only deal with the structured part. Some assumptions on the unstructured dynamics must be made beforehand. It is interesting that the frequency ranges are surprisingly similar for a wide range of aircrafts. The airframe typically has resonances at 40 rad/s, the nonstationary aerodynamics around 60 rad/s actuator dynamics is around 12 rad/s, presampling filters and delays around 100 rad/s.

It is concluded that adaptive control can help only with the structured model errors but that it must work in the presence of unstructured model uncertainties. Some knowledge about the unstructured model errors must be available. This knowledge must be included in the design of the adaptive system.

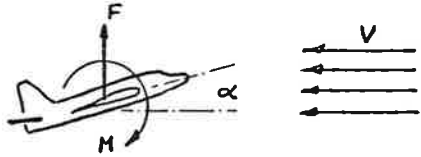
The main conclusion is that the adaptive control has lost the competition against gainscheduling for regular flight control. Some potential problems where adaptive control may apply are when airdata is impractical (small missiles and reentry vehicles), where air data will not work (flexure and flutter control).

It is suggested that some work in the adaptive field is devoted to understand the engineering fixes that are made to make the systems work and particularly that unstructured uncertainties are considered.

References

- [1] G. Stein (1980): Adaptive Flight Control: A Pragmatic View. In Narendra and Monopoli (eds) Applications of Adaptive Control. Academic Press, New York 1980.
- [2] IEEE Transaction on Automatic Control AC-22, Mini-issue on NASA's advanced control law program for the F-8 DFBW aircraft.

A LITTLE AIRPLANE PHYSICS

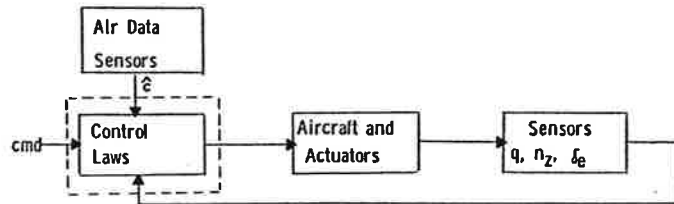


- FORCES/MOMENTS PROPORTIONAL TO DYNAMIC PRESSURE: $\bar{q} = \rho V^2 / 2$ 100:1
- FORCES/MOMENTS MOVE WITH MACH: $M = V/a$ 10:1
- SOME FORCE/MOMENT SLOPES CHANGE WITH ANGLE-OF-ATTACK: $\alpha = \tan^{-1} V_y/V_x$ 5:1

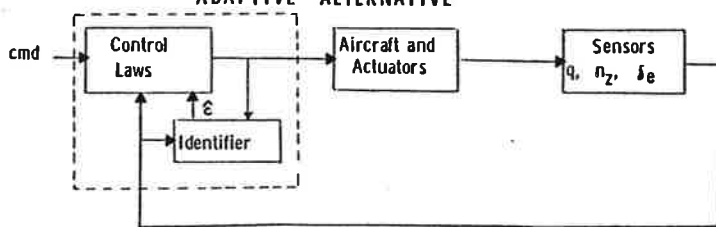
$$\dot{x} = A(\bar{q}, M, \alpha)x + B(\bar{q}, M, \alpha)u$$

COMPETING FLIGHT CONTROL ALTERNATIVES

CONVENTIONAL AIR-DATA-SCHEDULED CONTROL



ADAPTIVE ALTERNATIVE



HISTORY AND ISSUES
IN ADAPTIVE FLIGHT CONTROL

GUNTHER STEIN
NSF - STU
WORKSHOP ON ADAPTIVE CONTROL
JULY 1984

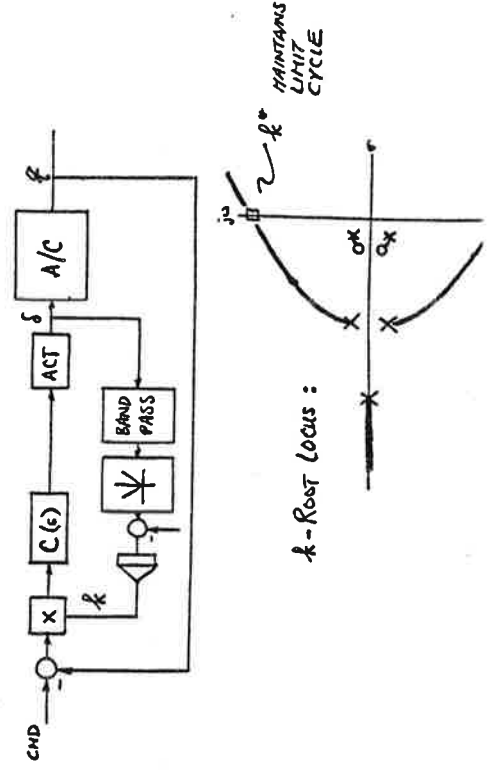
THREE QUESTIONS

- WHY ARE THERE NO ADAPTIVE FLIGHT CONTROL SYSTEMS ?
- WHY HAS ADAPTIVE THEORY BEEN IRRELEVANT TO ALL ATTEMPTED DESIGNS ?
- WHAT IS HAPPENING TO CHANGE THIS ?

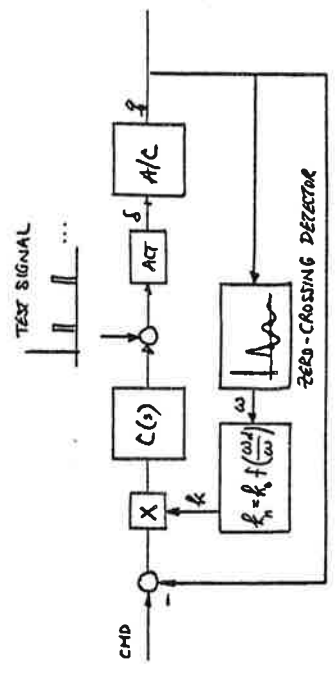
HISTORY OF COMPETITION

- 1960
 - ROOR AIRDATA SYSTEMS
 - SEVERAL ADAPIVE APPLICATIONS
 - K-15
 - F-111
 - F-101 TESTS
 - F-4 TESTS
 - ALL CERTAINTY EQUIVALENCE
- 1970
 - MUCH IMPROVED AIRDATA
 - NO ADAPIVE APPLICATIONS
 - 17EEN FIGHTERS SHUTTLE
 - RE-EVALUATION
 - REASONS
 - MODERN THEORY
 - DIGITAL IMPLEMENTATION
 - RELIABILITY
 - UNSOLVED PROBLEMS
 - NASA F-8C PROGRAM
 - DESIGN SIMULATOR EVAL
 - FLIGHT TEST
 - CONCEPTS TAC OCT77
 - MULTIPLE MODEL
 - MODEL REFERENCE
 - CERTAINTY EQUIVALENCE (MLE / LA)
 - FLUTTER TESTS ?
- 1980

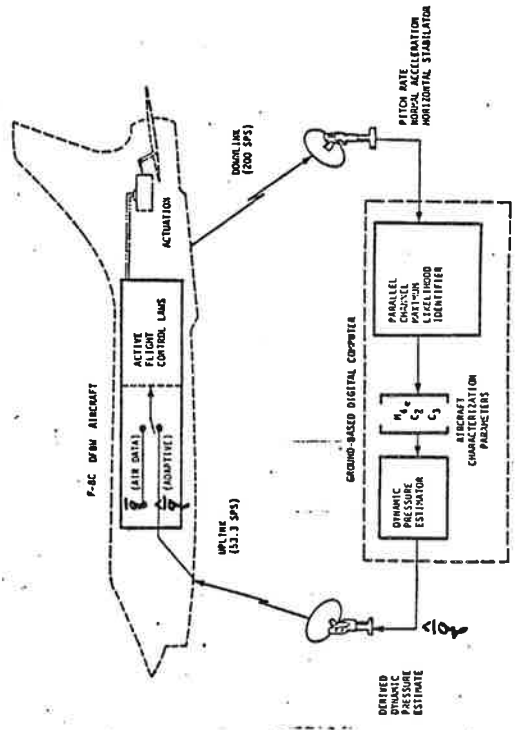
X-15 LIMIT CYCLE SYSTEM



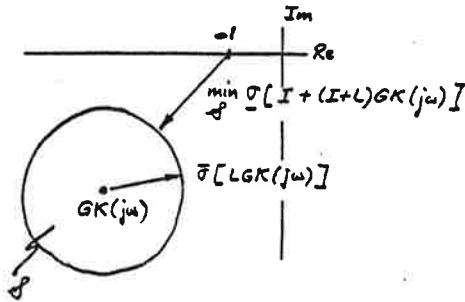
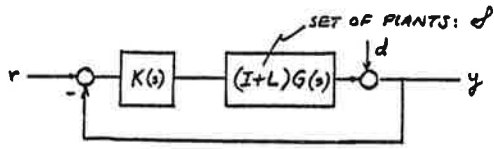
F-111 EXPLICIT IDENTIFIER



F-8C EXPLICIT IDENTIFIER



THE BASIC FEEDBACK PROBLEM



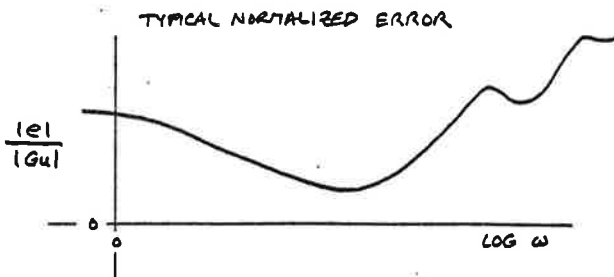
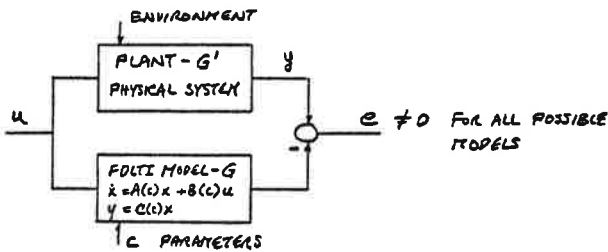
- FOR PERFORMANCE

$$\min_{\mathcal{P}} \sigma[I + (I+L)GK(j\omega)] \text{ LARGE AT SOME } \omega\text{'S}$$

- FOR STABILITY

$$\min_{\mathcal{P}} \sigma[I + (I+L)GK(j\omega)] \neq 0 \text{ AT ALL } \omega\text{'S}$$

UNCERTAINTIES IN FDLE MODELS



RESULTING FUNDAMENTAL "FACT OF LIFE"

GOOD FEEDBACK PERFORMANCE

(IE. $GK \gg I$ OVER A GIVEN FREQUENCY RANGE)

IS POSSIBLE IF AND ONLY IF

MODEL UNCERTAINTIES ARE SUFFICIENTLY SMALL

(IE. $\bar{\sigma}[L] < 1$ IN THE SAME FREQUENCY RANGE)

WE HAVE TWO OPTIONS

- KNOW OUR PLANT BEFORE HAND, OR
- LEARN IT AS WE GO

TWO TYPES OF ERRORS

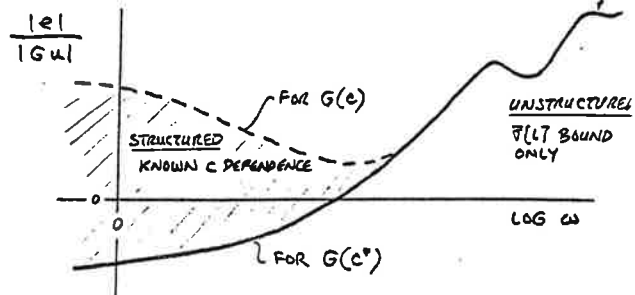
STRUCTURED - ERRORS WHICH CAN BE ELIMINATED BY ADJUSTING PARAMETERS c TO BEST MATCH THE PLANT

UNSTRUCTURED - ERRORS WHICH REMAIN

$$e = G'u - G(c^*)u$$

$$= (I+L)Gu - Gu$$

$$= LGu \Rightarrow \bar{\sigma}[L] = \max_u \frac{|e|}{|G(c^*)u|}$$

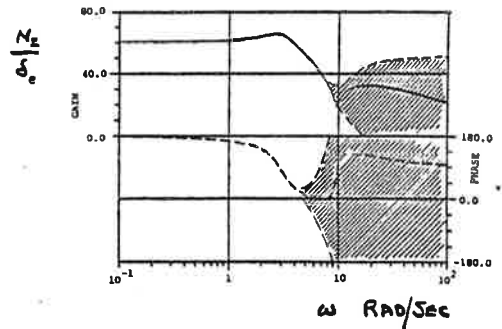


- SOME TECHNIQUES:
- NEGLECTED DYNAMICS
 - ACTUATOR SERVOS
 - RESONANCES
 - DIFFUSIONS
 - TIME DELAYS
 - CONE-BOUNDED NONLINEAR
 - ETC

MORE "FACTS OF LIFE"

- ADAPTIVE CONTROL CAN HELP WITH STRUCTURED ERRORS ONLY
- ADAPTIVE CONTROL MUST WORK IN THE PRESENCE OF UNSTRUCTURED ERRORS
- WE MUST KNOW (OR ASSUME) THE $\bar{\sigma}[L]$ CURVE BEFORE HAND
 - FREQ RANGE OVER WHICH GK CAN BE LARGE
 - BOUNDS ON GK OUTSIDE THAT RANGE

F-8C UNSTRUCTURED UNCERTAINTIES



- ACTUATOR SERVOS ~ 12 R/S
- AIRFRAME STRUCTURAL ~ 40 R/S RESONANCES
- UNSTEADY AERODYNAMICS ~ 60 R/S
- SAMPLING DELAYS / PREFILTERS ~ 100

ADAPTIVE CONTROL TASKS

TASK 1

LEARN C^* (EXPLICITLY OR IMPLICITLY)

TASK 2

IMPLEMENT CONTROL LAW WITH GK CONSISTENT WITH $\bar{\sigma}[L]$ CURVE AND WITH EXPECTED C^* ERRORS

TASK 3

PROTECT AGAINST UNSTRUCTURED ERRORS

IE. DON'T LET THEM CONFUSE THE LEARNING PROCESS

DON'T MAKE GK LARGE WHERE $\bar{\sigma}[L] > 1$

ADAPTIVE CONTROL SOLUTIONS

VIA STOCHASTIC OPTIMIZATION

OPTIMIZATION PROBLEM

$$\text{GIVEN } \dot{x} = A(c)x + B(c)u + \xi$$

$$y = C(c)x + \eta$$

$$\text{FIND } u = F\{y(\tau), \tau \leq t\}$$

TO MINIMIZE

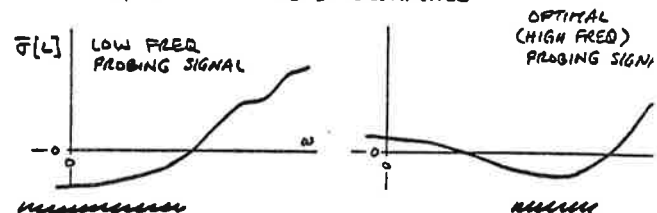
$$J = E_{\xi, \eta, c} \left\{ \frac{1}{T} \int_0^T L(x, u) dt \right\} \quad T \rightarrow \infty$$

FEATURES :

THEORETICALLY OPTIMAL

- GENERALLY UNSOLVABLE
- TASKS 1 & 2 ARE OPTIMALLY BLENDED (DUAL EFFECT)
- TASK 3 IS NOT ADDRESSED

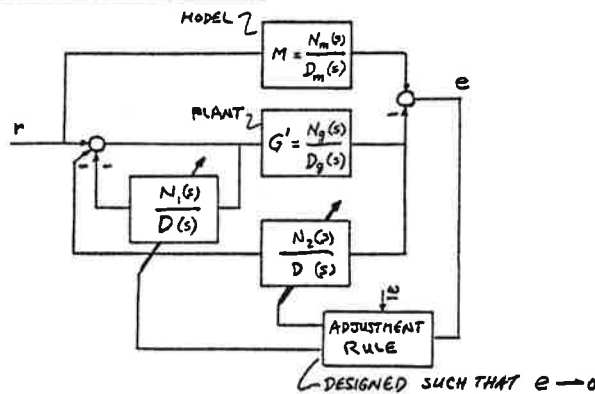
SOLUTIONS ARE (PROBABLY) EASILY CONFOUNDED BY UNSTRUCTURED UNCERTAINTIES



ADAPTIVE CONTROL SOLUTIONS
VIA STABILITY THEORY

ASSUMPTIONS REQUIRED
FOR STABILITY/CONVERGENCE PROOFS

MODEL REFERENCE STRUCTURE



PLANT: $G'(s) = \frac{N_g(s)}{D_g(s)} \rightarrow \frac{k}{s^m}$ AS $|s| \rightarrow \infty$

ASSUMPTIONS:

- (1) $N_g(s)$ IS STABLE
- (2) $D_g(s)$ HAS KNOWN MAX DEGREE
- (3) k HAS KNOWN SIGN
- (4) RELATIVE DEGREE m IS KNOWN

IMPLICATIONS:

- ZERO UNSTRUCTURED UNCERTAINTIES
- HIGH GAIN FEEDBACK WITHOUT ADAPTATION WILL ACHIEVE MODEL FOLLOWING

FEATURES: THEORETICALLY GLOBALLY STABLE

- TASK 1 ACHIEVED IMPLICITLY BY ADJUSTING N_1, N_2, D TO CANCEL PLANT

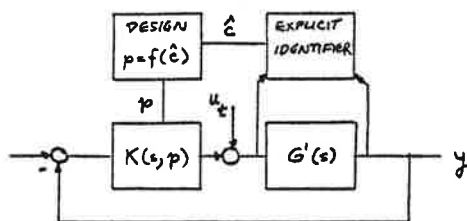
$$\frac{N_g D}{D_g(N_1 + D) + N_g N_2} = \frac{N_m}{D_m}$$

- TASK 2 ACHIEVED BY APPROPRIATE SELECTION OF $M(s)$
- TASK 3 NOT ADDRESSED

SOLUTIONS KNOWN TO BE CONFOUNDED BY UNSTRUCTURED ERROR

ADAPTIVE CONTROL SOLUTIONS
VIA CERTAINTY EQUIVALENCE

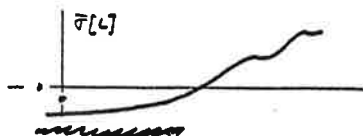
F-BC PROGRAM CONCLUSIONS



FEATURES: NO THEORETICAL PROPERTIES

- TASK 1: ACHIEVED BY EXPLICIT IDENTIFICATION OF 'BEST FIT' c VECTOR
- TASK 2: ACHIEVED BY PRE-DESIGNED COMPENSATOR $K(s, p)$ WITH $p = f(\hat{c})$
- TASK 3: ACHIEVED BY

- 'BEST FIT' FUNCTION AND TEST SIGNAL u_f CONFINED TO DESIRED FREQ RANGE



- $K(\cdot)$ AND $f(\cdot)$ CONSTRAINED TO MAKE GK LARGE ONLY AS ALLOWED BY $\bar{v}[L]$

- ADAPTIVE FLIGHT CONTROLS STILL CAN'T BEAT AIRDATA

- MOST 'MODERN' CONCEPTS DID NOT WARRANT FLIGHT TESTING
- THE MLE CONCEPT BARELY MATCHES AIRDATA PERFORMANCE

- ADAPTIVE FLIGHT CONTROLS NEED TEST SIGNALS (SUFFICIENTLY RICH INPUTS)
- PILOTS HATE THESE
- ADAPTIVE FLIGHT CONTROL DESIGNERS SHOULD TURN TO PROBLEMS BEYOND THE CAPABILITY OF AIRDATA

POTENTIAL ADAPTIVE PROBLEMS

STATUS

WHERE AIRDATA IS IMPRACTICAL

- SMALL MISSILES
- REENTRY

ADAPT SYSTEMS EXIST
 BLENDED AERO/INERTIAL DATA WINS OUT

WHERE AIRDATA WON'T WORK

- DIRECT FORCE MODES
- FLEXURE/FLUTTER CONTROL

IN-FLIGHT CAL WINS OUT
 WIND TUNNEL TESTS LOOK GOOD

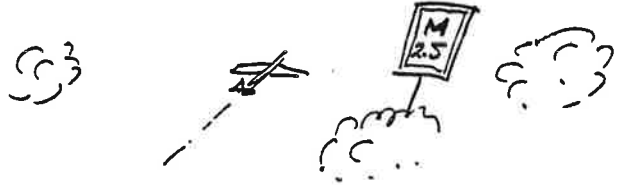
PIE-IN-THE-SKY

- SURFACE RECONFIGURATION
- LARGE SPACE STRUCTURES

DHYB

FLEXURE/FLUTTER CONTROL

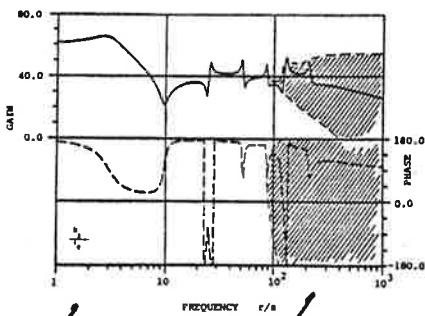
- AIRPLANES HAVE SPEED LIMITS



- ACTIVE CONTROL CAN EXTEND LIMITS BY AUGMENTING AERDELASTIC DAMPING

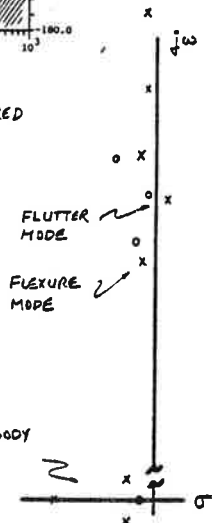
- DYNAMICS BETWEEN 10-1000 R/SEC
- HIGHLY DETAILED, POORLY KNOWN
- NO VIABLE CONTROL SOLUTIONS TODAY

FLEXURE/FLUTTER MODELS



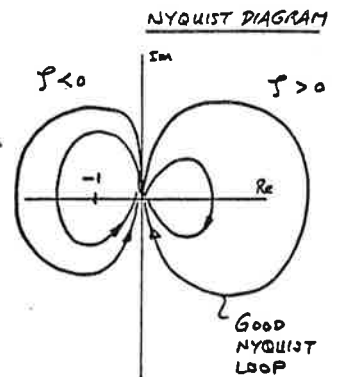
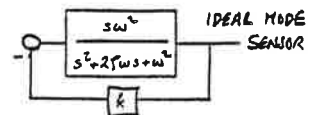
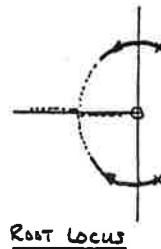
RIGID BODY AND STRUCTURAL RESONANCE DYNAMICS

UNSTRUCTURED ERRORS

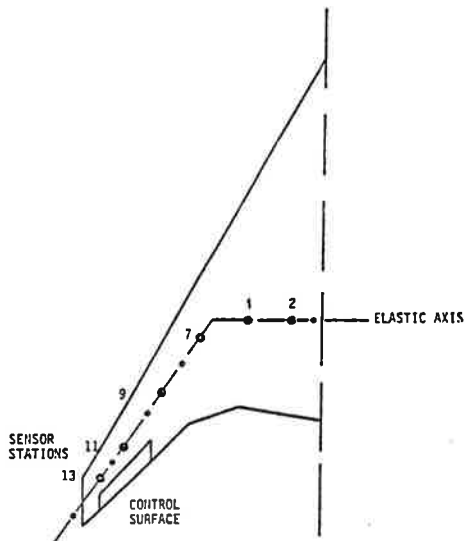


IDEALLY FLEXURE/FLUTTER CONTROL IS EASY

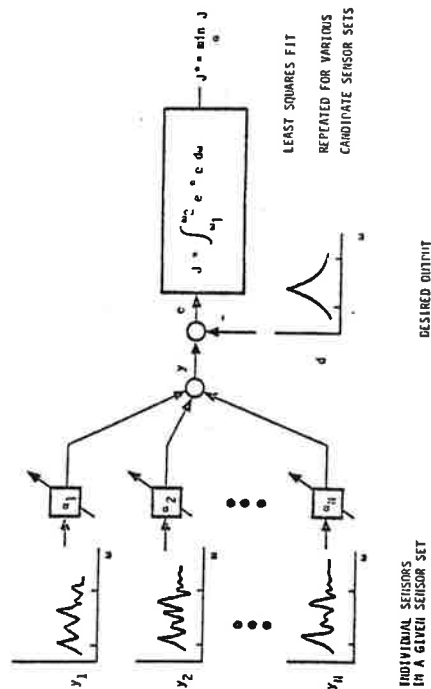
- SENSE EACH RESONANCE INDIVIDUALLY
- CLOSE SIMPLE RATE LOOPS AROUND EACH MODE



IDEAL MODE SENSORS ARE SYNTHESIZED FROM VARIOUS REAL SENSORS



IDEAL SENSOR SYNTHESIS



ADAPTATION IN FLUTTER CONTROL ?

- 1) FLUTTER DETECTION
 - IS A MODE UNSTABLE ?
 - IF SO, WHICH ONE ?
- 2) CONTROL LAW DESIGN
 - GAIN & PHASE COMPENSATION WITH GOOD SENSOR SYNTHESIS AVAILABLE
- 3) ON-LINE SENSOR SYNTHESIS

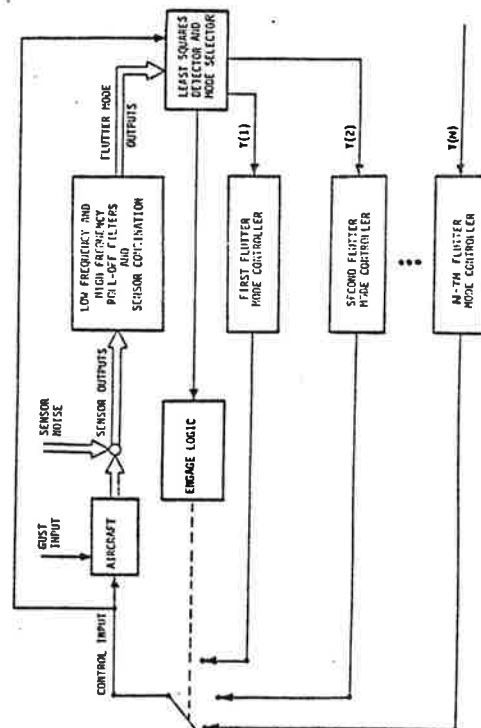
MAIN ISSUE

SPEED OF RESPONSE

STATE OF THE ART

- STEPS (1) & (2) FEASIBLE IN SIMULATIONS AND WIND TUNNEL TESTS
- STEP (3) IS CURRENT RESEARCH

SIMPLE ADAPTIVE FLUTTER CONTROL



RAW ACCELEROMETERS

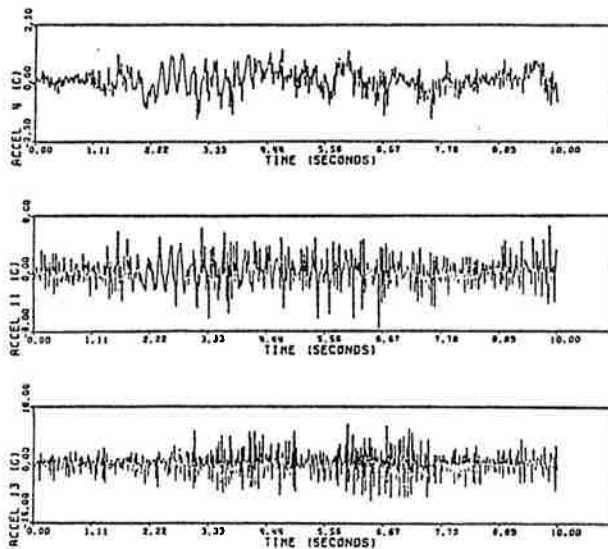
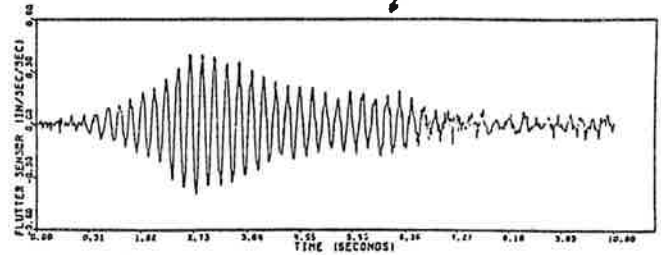


Figure 33. Transient Responses for Run 7 of Table 13 (MLB, Case 1, 1.3 V_r) (continued)

IDEAL MODE SENSOR 7



CONTROL INPUT 7

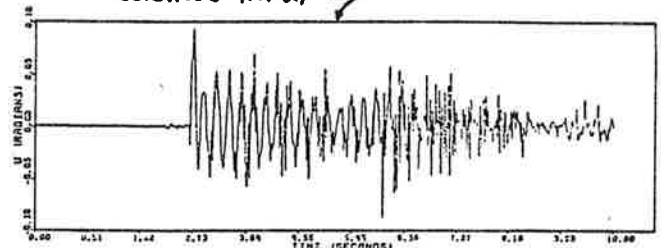


Figure 33. Transient Responses for Run 7 of Table 13 (MLB, Case 1, 1.3 V_r) (continued)

PROGRESS IN ADAPTIVE THEORY

- OLD ENGINEERING FIXES GET SERIOUS THEORETICAL ATTENTION
 - DEAD ZONES Peterson, Orlicki
 - RETARDATION Kreisselmeier, Ioannou, Kaufman
 - SAMPLE RATES Astrom et al., Rohrs
- NEW APPLICATION OF OLD THEORETICAL TOOLS
 - AVERAGING ANALYSES Astrom, Krause
- REFORMULATIONS
 - INCLUSION OF UNSTRUCTURED UNCERTAINTIES Kosut, Johnson

CONCLUSIONS

BAD NEWS

WE LOST THE COMPETITION TO SENSOR BUILDERS FOR ALL CONVENTIONAL FLIGHT CONTROL PROBLEMS

GOOD NEWS

THERE ARE A FEW PROBLEMS WHERE OUR TECHNOLOGY OFFERS HOPE

BAD NEWS

WE CAN'T SOLVE THESE PROBLEMS (YET)

GOOD NEWS

THE THEORY IS COMING ALONG TO HELP !

Self tuning control of the dissolved oxygen concentration in activated sludge systems

Gustaf Olsson and Lars Rundqwist

Department of Automatic Control
Lund Institute of Technology
Lund, Sweden

The activated sludge process is recognized as the most common and major unit process for the reduction of organic waste. An overview of of the control problems is found in [1].

A fully structured model of an activated sludge system is very complex and includes the following phenomena,

- * the degradation of degradable pollutants, containing both organic carbon, phosphorus and nitrogen;
- * cell growth and basal metabolism;
- * the oxygen requirements of the system;
- * the flow regime in the aeration basin;
- * the representation of the settler and clarifier performance;
- * the effect of secondary parameters, such as temperature, pH and toxic or inhibitory substances.

The goal for the reactor operation is of course to degrade the degradable pollutants. However, the operation must be such that organisms with the preferred floc formation are produced, thus giving desired clarification and thickening properties. Otherwise the operation of the system will fail, even if the degradation is efficient.

The DO concentration is an essential variable of the activated sludge process. It has a significant influence on both the plant operation economy and on the biological activity, and consequently on the quality of the effluent water.

A detailed derivation of the equations can be found in [1]. In a dispersed plug flow reactor the resulting DO dynamics can be described by

$$\frac{\partial c}{\partial t} = E \frac{\partial^2 c}{\partial z^2} - v \frac{\partial c}{\partial z} + k_L a (c^s - c) - R \quad (1)$$

where

z =length along the reactor

$c(z,t)$ =dissolved oxygen concentration

c^S =dissolved oxygen saturation concentration

$k_L a$ =oxygen mass transfer rate= $f(u)$, u = air flow rate

E =dispersion coefficient

v =stream velocity

R =oxygen uptake rate (respiration rate)

The phenomena influencing the DO concentration have widely different timescales.

The control of DO as a physical variable does not require any in-depth knowledge of the microbial dynamics. The problem of finding the right DO set-point has been discussed e.g. in [3].

There are some important reasons for self-tuning control. The oxygen transfer rate is approximately proportional to the control signal, the air flow rate. Therefore a self-tuner can compensate for different "time constants" at different operating levels. Moreover, the oxygen transfer rate is time varying on a day-to-day time scale. The respiration R varies on an hourly time scale and is the main reason for control. However, R is interesting to know for other reasons. This can be part of the estimation scheme of a self-tuner. If the fact is used, that $k_L a$ and R vary in different time scales they can be estimated simultaneously.

Since last year full scale experiments of DO control have been performed at the Käppala wastewater treatment plant at Lidingö, outside Stockholm. The plant serves the northern suburbs of Stockholm and has a flow rate of about 2-4 m³/sec. A Novatune controller has been installed to take care of both the DO control loop and the air production system.

The figure shows some important features of the control. A constant setpoint of the DO concentration is given to the STR, sampled every 10 minutes. The DO sensor is fed into the controller and the control signal is cascaded with a local analog controller to adjust a throttle valve for the air flow rate. A limiting switch tells the STR if the valve saturates.

The pressure control is currently based on constant pressure set-point. It is kept via guide vanes on 3 of the 6 compressors. This does not give full control authority, and discontinuities of the control signals cannot be avoided. A pressure optimized will be added as soon as a sensor of the throttle valve angle can be measured. Then the pressure will be minimized so as to keep the valve as open as possible.

The results hitherto are encouraging but the evaluations of the biological properties due to control have just started. An expansion of the control to the whole activated sludge system will be made during the Fall of 1984.

References

- Olsson, G. State of the Art in Sewage Treatment Plant Control. AICHE Symp Series, 72, 52-76, 1977.
- Briggs R. and G. Olsson (1982) On-line Measurement, Data Processing and Digital Control, The Institute of Water Pollution Control, Symp. Data Processing in the Water Industry, Bournemouth, England, 20-21 April 1982.
- Olsson, G. (1980), Modeling and Control of the Activated Sludge Process, Invited paper, American Chemical Society Symp. Series no 124, Computer Applications to Chemical Engineering, R.G. Squires and G.V. Reklaitis, Eds.
- Olsson, G. (1979) Automatic Control in Wastewater Treatment Plants, Invited Paper, Int. Environmental Colloquium, Liege, Belgium, May 1979.

C O N T E N T

SELF TUNING CONTROL OF THE
DISSOLVED OXYGEN CONCENTRATION
IN ACTIVATED SLUDGE SYSTEMS

GUSTAF OLSSON

LARS RUNDQWIST

→ INTRODUCTION

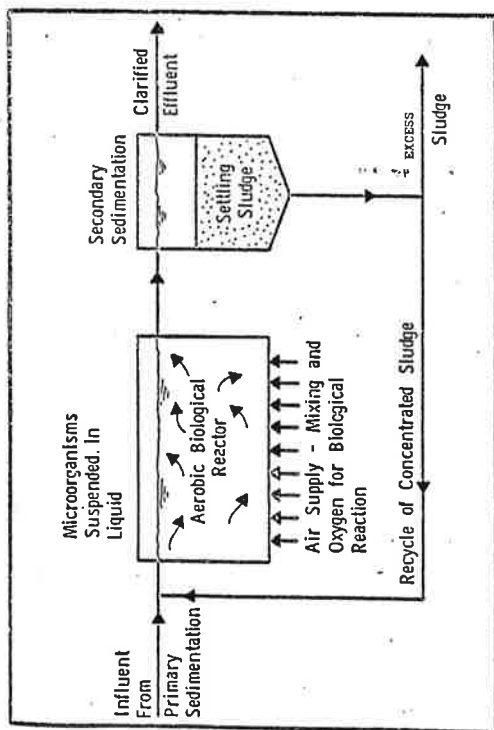
DISSOLVED OXYGEN DYNAMICS

WHY SELF TUNING CONTROL ?

THE KAPPALA PLANT

EVALUATION OF RESULTS

THE ACTIVATED SLUDGE PROCESS



THE ACTIVATED SLUDGE PROCESS

INFLUENT WATER CONTAINS

BIODEGRADABLE POLLUTANTS

NON_BIODEGRADABLE POLLUTANTS

CHEMICALS

TOXIC MATERIAL

INERT MATERIAL

MICROORGANISMS REACT WITH POLLUTANTS AND OXYGEN

TO FORM MORE CELL MASS

CARBON DIOXIDE

WATER

OXYGEN IS SUPPLIED FROM DIFFUSERS

INCENTIVES FOR CONTROL

- RISING OPERATING COSTS (600 % BETWEEN 1971 AND 1984)

 - ENERGY (AIR SUPPLY, PUMPING)

 - CHEMICALS

 - PERSONNEL

- STRICTER EFFLUENT CONTROL

- TIME VARYING INFLUENT

 - HYDRAULICS

 - CONCENTRATION

 - COMPOSITION

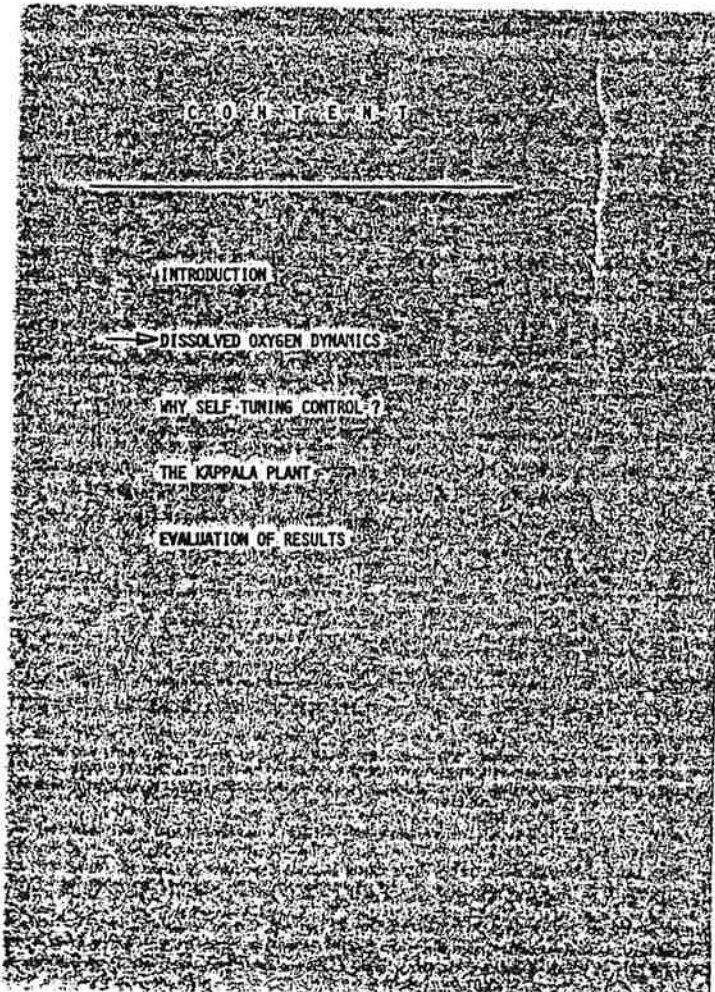
POSITIVE DRIVING FORCES

- BETTER PROCESS KNOWLEDGE

- IMPROVED INSTRUMENTS

- COMPUTER COSTS AND COMPUTER PERFORMANCE

- BETTER CONTROL METHODS



THE DISSOLVED OXYGEN CONCENTRATION

ECONOMY - MINIMIZE THE CONCENTRATION

SET POINT - FORMATION OF DIFFERENT ORGANISMS

- LIMITATION OF GROWTH

MIXING - DO CONCENTRATION AND MIXING ARE COUPLED

- FLOC FORMATION

MULTI REACTOR SYSTEM - ANOXIC AND OXIC ZONES IN SERIES

PROFILES OF DO - CAN GENERALLY NOT CONTROL THE PROFILE ALONG THE REACTOR

DISSOLVED OXYGEN DYNAMICS

IN COMPLETE MIX REACTOR - MASS BALANCE

$$\frac{dc}{dt} = \frac{Q_{in}}{V} \cdot c_{in} - \frac{Q_{out}}{V} \cdot c + \underbrace{\alpha \cdot u (c^* - c)}_{\text{OXYGEN TRANSFER}} - \underbrace{R}_{\text{RESPIRATION}}$$

OXYGEN TRANSFER

α VARIES SLOWLY (DAYS - WEEKS)

RESPIRATION

R DEPENDS ON BIOLOGICAL GROWTH AND MAINTENANCE
VARIES QUICKLY (MINUTES - HOURS)

WHY SELF-TUNING CONTROL OF DO?

- BILINEAR SYSTEM - "TIME CONSTANT" VARIES WITH OPERATING LEVEL
- α VARIES SLOWLY
- WANT TO KNOW THE RESPIRATION FOR OTHER PURPOSES, ESTIMATE FROM THE MASS BALANCE!
- FEED FORWARD SIGNALS CAN BE INTRODUCED FROM INFLUENT (COD, TOC)

CONTENTS

INTRODUCTION

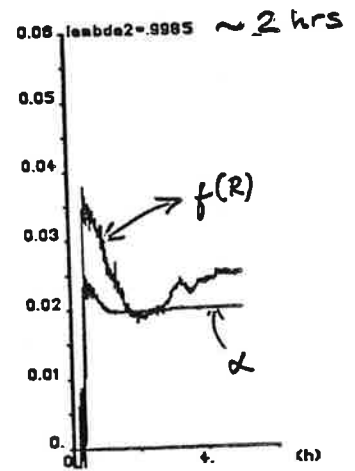
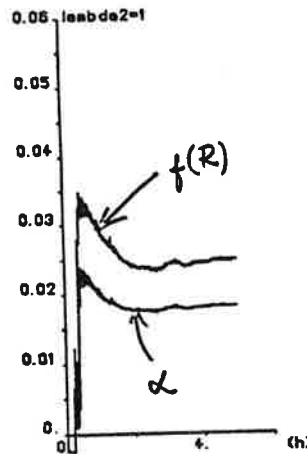
DISSOLVED OXYGEN DYNAMICS

→ WHY SELF TUNING CONTROL ?

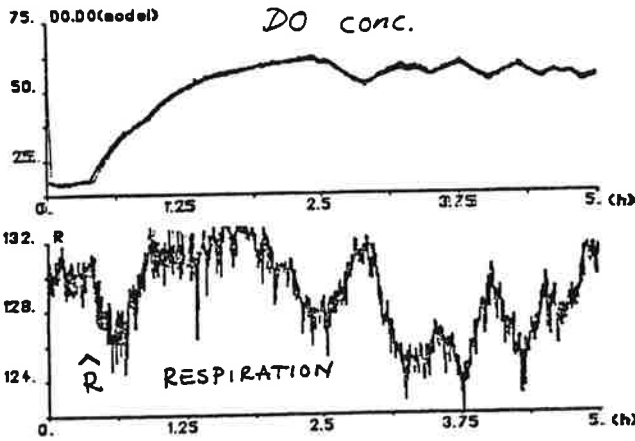
THE KAPPALA PLANT

EVALUATION OF RESULTS

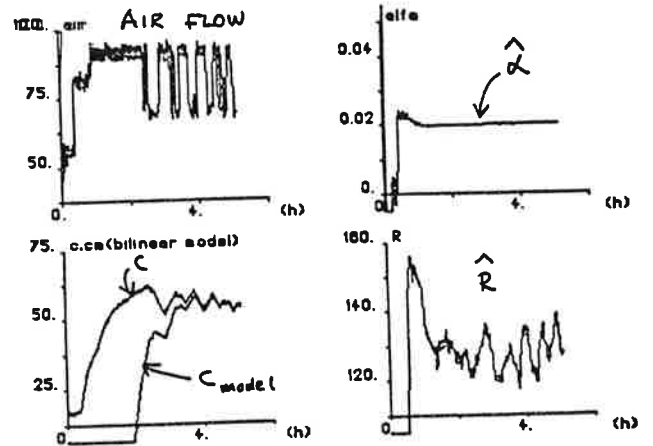
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C O N T E N T

INTRODUCTION

DISSOLVED OXYGEN DYNAMICS

WHY SELF TUNING CONTROL ?

☞ THE KAPPALA PLANT

EVALUATION OF RESULTS

THE KAPPALA PLANT

- SERVES THE NORTHERN PART OF STOCKHOLM
- ABOUT 500 000 PERSONS
- INFLUENT FLOW RATE 2-4 m³/SEC
- SIX PARALLELL BASINS
- ELECTRIC BILL ABOUT 1 MKR/YEAR

C O N T E N T

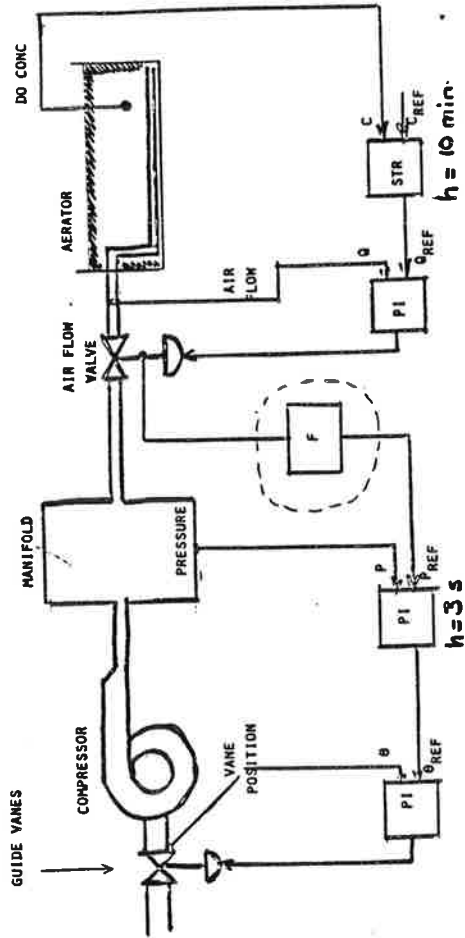
INTRODUCTION

DISSOLVED OXYGEN DYNAMICS

WHY SELF TUNING CONTROL ?

THE KAPPALA PLANT

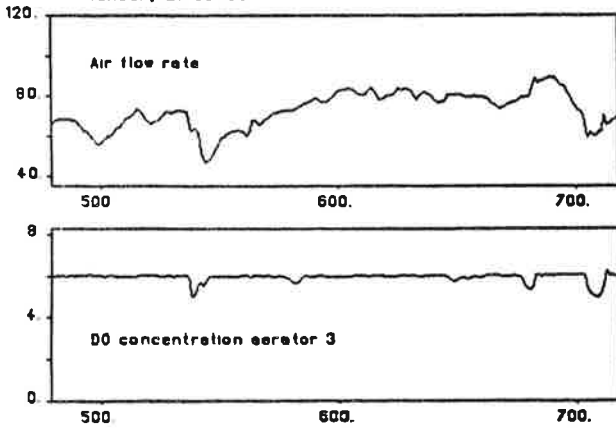
➔ EVALUATION OF RESULTS



DO CONTROL AT KAPPALA

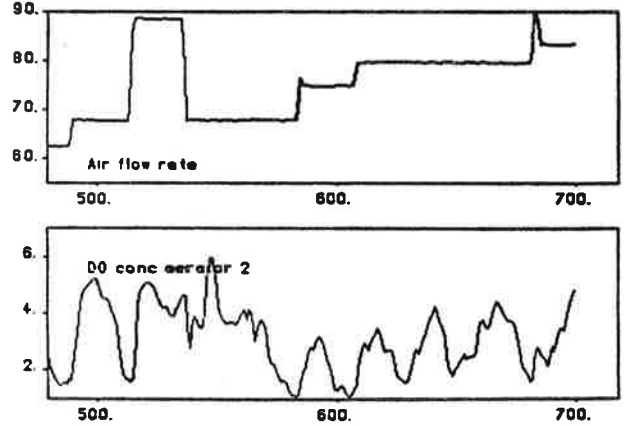
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January 21-30 1984

SELF TUNING CONTROL



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MANUAL CONTROL



EVALUATION OF RESULTS

- CONTROL AUTHORITY
 - DISCONTINUITIES WHEN BLOWERS SWITCH ON/OFF
 - CERTAIN RANGES DIFFICULT
- THROTTLE VALVE
 - POSITION HAS TO BE MEASURED
- FINDING THE PROPER PRESSURE
- WATER QUALITY
 - BEING EVALUATED BY INSTITUTE OF SURFACE CHEMISTRY
- DETECTION OF TOXIC INPUTS
- EXPANSION TO ALL SIX REACTORS

Adaptive pole placement for robots and servomechanisms

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Robots or mechanical manipulators are multi-degree of freedom systems whose dynamics are often nonlinear. Furthermore, because such systems pick up and move loads of various shapes and weights, the dynamics can also be viewed as time varying. The difficulties of working with nonlinear dynamics and time variation lead researchers to consider adaptive control as an alternative. Many recent papers have applied adaptive control schemes originally developed for unknown linear dynamics to such systems[1]-[4]. In this presentation we will begin by showing that this is a potentially dangerous design methodology. That is, adaptive controllers of this type will not perform as expected if the nonlinearities are dominant.

With this as motivation, we then develop a new approach to adaptive control of manipulators where the controller adapts to load changes but not nonlinearities. Rather the nonlinearities are included in design of the adaptive control law. The approach first uses nonlinear feedback to cancel the effect of the nonlinearities. Linear feedback is then used to place closed loop poles. The controller is intended for digital computer implementation and is discrete in nature. Most importantly, the design is based upon discrete time model for the nonlinear dynamics which is derived using Euler approximations for derivatives.

As a step toward understanding the stability and performance properties of such an adaptive scheme, one must first determine the stability and performance properties of the corresponding fixed computer control algorithm designed using the same Euler approximation model. To this end, we restrict our attention to the case of a single link or single degree of freedom manipulator. Assuming torque to be supplied by a D.C. motor, this boils down to design of a D.C. servo system. However, because the link represents an asymmetric load rotating in a gravitational field the resulting model is still nonlinear. In the case where the link rotates in a horizontal plane the model becomes linear. By considering this simple case of computer control of a linear servo, it is shown that fixed computer control algorithms for pole placement, designed using Euler approximation models are stable for a very wide range of model parameters, sample rates and closed pole locations.

Furthermore, it is shown that considerable performance improvements can be obtained when designing zero cancelling controllers such as are used in model matching. In particular the problems which arise because sample data models have zeros on or near the unit circle can be avoided. Since many adaptive control schemes are based upon fixed zero cancelling control strategies these results are also of potential importance in the design of adaptive sample data systems.

During the course of the presentation, simulations will be presented

demonstrating the potential performance of the new nonlinear adaptive control algorithm on a three degree of freedom manipulator. In addition experimental results will be presented for a hardware implementation on a one degree of freedom servo system in both the linear and nonlinear cases.

References

- S Dubowsky and D.T. Desforges, "The Application of Model Reference Adaptive Control to Robotic Manipulators," Journal of Dynamic System Measurement and Control, September, 1979, pp.193-200.
- A.J. Koivo and T.H. Gou, "Adaptive control of Robotic Manipulators," IEEE Trans. on A.C., Vol. AC-28, No. 2, February 1983
- C.S.G. Lee and M.J. Chung, "An Adaptive Control Strategy for Computer-Based Manipulators," Proceedings 21stCDC, Orlando, December, 1982, pp. 95-100.
- C. Chung and G. Leninger, "Adaptive Self-tuning Control of Manipulators in Task Coordinate System," Proceedings IEEE Comp. Soc. Int. Conf. on Robotics, March, 1984, Atlanta, pp. 546-555.
- H. Elliott, T. Depkovich, J. Kelly, B. Draper, "Nonlinear Adaptive Control of Mechanical Linkage Systems with Application to Robotics," Proceedings of 1983 ACC, June, 1983, San Francisco, pp. 1050-1055.
- G.C. Goodwin and K.S. Sin, Adaptive Filtering Prediction and Control, Prentice Hall, Englewood Cliffs, 1984.

ADAPTIVE POLE PLACEMENT FOR ROBOTS AND SERVO MECHANISMS

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OUTLINE OF PRESENTATION

- I. INTRODUCTION
- II. LINEAR VS NONLINEAR ADAPTIVE CONTROL
Case Study - 1 DOF
- III. GENERAL ALGORITHM FOR CONTROL
OF MANIPULATORS
2 DOF Case
- IV. 3 DOF Case and Engineering
simplifications
- V. ISSUES IN Stability Analysis
- VI
- VII. Experimental Results

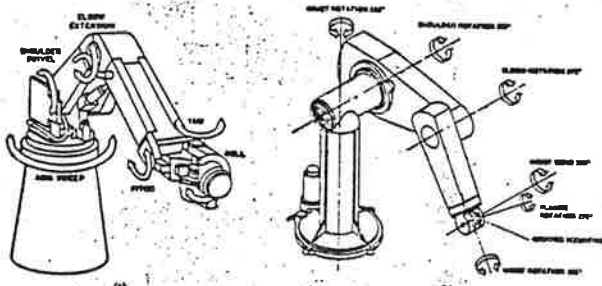


Fig. 3. Examples of industrial robots. (a) Cincinnati Milacron T3, (b) Unimation PUMA 602. Courtesy of UNIMATION, Inc.

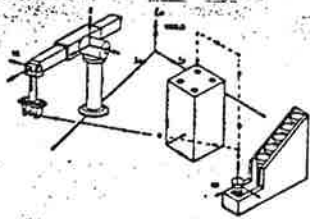
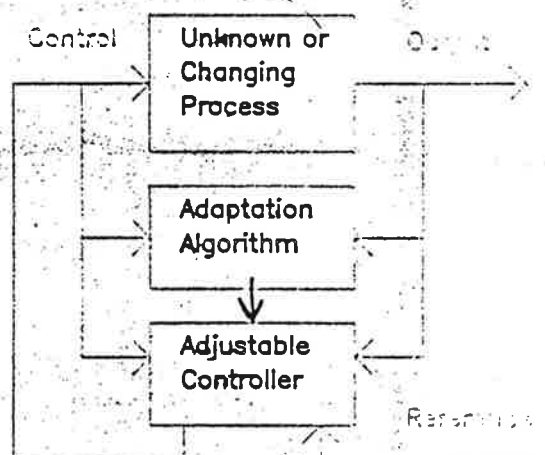


Fig. 4. Simple robot used for simulation.

Problems In Manipulator Control

- * Changing Dynamics
 - load changes
 - environmental changes
- * Nonlinearities
 - coordinate transformations
 - dynamics

Why Adaptive Control?



* Changing Dynamics

~~* Nonlinearities~~

General Form of Nonlinear Dynamics

$$J(\dot{y}) + K\dot{y} + G(y) + F(y) = u$$

y = vector of joint angles

u = vector of motor torques

$J(y)$ = inertia matrix

K = damping matrix

$G(y, \dot{y})$ = Coriolis and centrifugal terms

$F(y)$ = gravitational terms

Control Problem

Find $u(t)$ so that tracks reference trajectory $y^*(t)$

General Approach

Let u have discrete and continuous components:

$$u(t) = u_d(t) + u_c(t)$$

$u_d(t)$ = adaptively computed nonlinearities

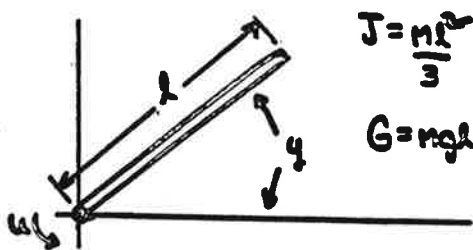
$u_c(t)$ = adaptive implementation of linear control

Advantages

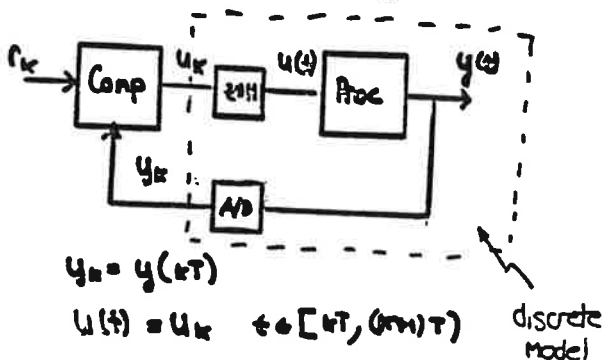
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II. Comparison of Two Approaches

Single-Link Case



$$J\ddot{y} + K\dot{y} + G\cos y = u$$



$$\dot{y} \approx (y_k - y_{k-1}) / T$$

$$\ddot{y} \approx (y_{k+1} - 2y_k + y_{k-1}) / T^2$$

$$\frac{J}{T^2} (y_{k+1} - 2y_k + y_{k-1}) + \frac{K}{T} (y_k - y_{k-1}) + G \cos y_k = u$$

$$\boxed{\Phi_k^T \Theta = u_k} \quad \text{Nonlinear Model}$$

$$\Phi_k^T = [y_{k+1}, y_k, y_{k-1}, \cos y_k]$$

$$\Theta^T = [\frac{J}{T^2}, \frac{K}{T} - \frac{2J}{T^2}, \frac{J}{T^2} - \frac{K}{T}, G]$$

$$\boxed{\tilde{\Phi}_k^T \tilde{\Theta} = u_k} \quad \text{Linearized Model}$$

$$\tilde{\Phi}_k^T = [y_{k+1}, y_k, y_{k-1}, u_{k-1}]$$

$$\tilde{\Theta} = ?$$

Deadbeat (Minimum Variance) Control

Assume y_k^* = seq. to be tracked
 know one step ahead
 ($r_k = y_{k+1}^*$)

Choose

$$u_k = \Phi_k^{*T} \theta \quad k \geq 0$$

$$\Phi_k^{*T} = [y_{k+1}^*, y_k, y_{k-1}, \cos y_k]$$

Guarantees $y_k = y_k^* \quad k \geq 1$

For linearized model

$$u_k = \tilde{\Phi}_k^{*T} \tilde{\theta}$$

$$\tilde{\Phi}_k^{*T} = [y_{k+1}^*, y_k, y_{k-1}, u_k]$$

Adaptive Implementation

$$u_k = \hat{\Phi}_k^{*T} \hat{\theta}_k$$

$$\hat{\theta}_k = \text{Est}(\theta)$$

$$u_k = \tilde{\hat{\Phi}}_k^{*T} \tilde{\hat{\theta}}_k$$

$$\tilde{\hat{\theta}}_k = \text{Est}(\tilde{\theta})$$

Least Squares Estimation

$$e_k = u_k - \hat{\Phi}_k^T \hat{\theta}_k \quad (o = u_k - \hat{\Phi}_k^T \theta)$$

$$\hat{\theta}_k = \hat{\theta}_{k-1} + \frac{P_{k-2} \hat{\Phi}_{k-1}^T e_k}{\lambda + \hat{\Phi}_{k-1}^T P_{k-2} \hat{\Phi}_{k-1}}$$

$$P_{k-1} = \left[P_{k-2} + \frac{P_{k-2} \hat{\Phi}_{k-1}^T \hat{\Phi}_{k-1} P_{k-2}}{\lambda + \hat{\Phi}_{k-1}^T P_{k-2} \hat{\Phi}_{k-1}} \right]^{-1}$$

$$P_1 = rI \quad r > 0$$

$0 < \lambda \leq 1$ (exponential weighting)

linear case $\hat{\Phi}_k, \hat{\theta}_k \rightarrow \tilde{\hat{\Phi}}_k, \tilde{\hat{\theta}}_k$

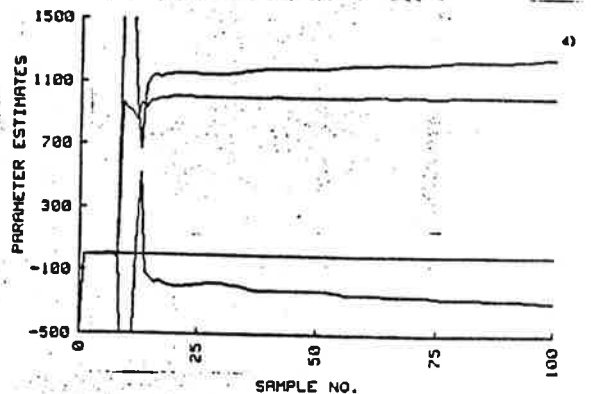
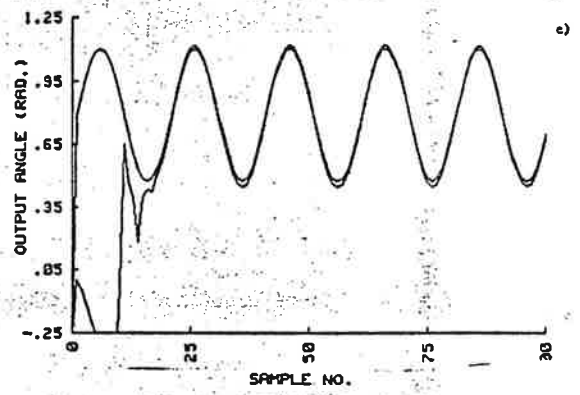
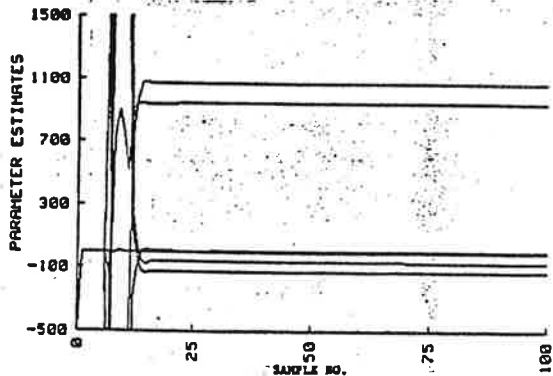
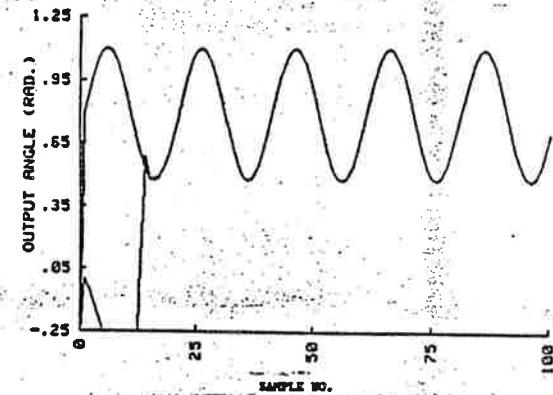


Figure 2. Response of nonlinear, a), b), and linear c), d) when $m=2, n=2$.

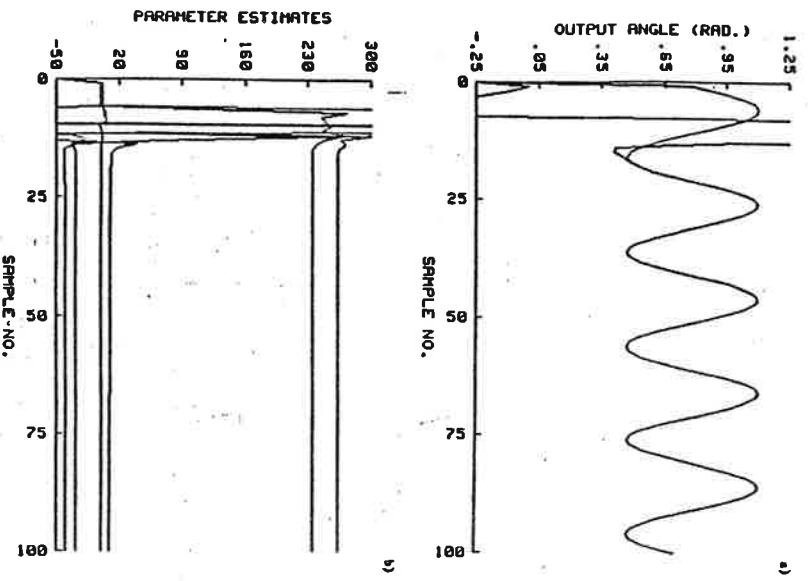


Figure 3. Dependence of output angle α), β), and linear c), d) when $w=2$, $l=1$.

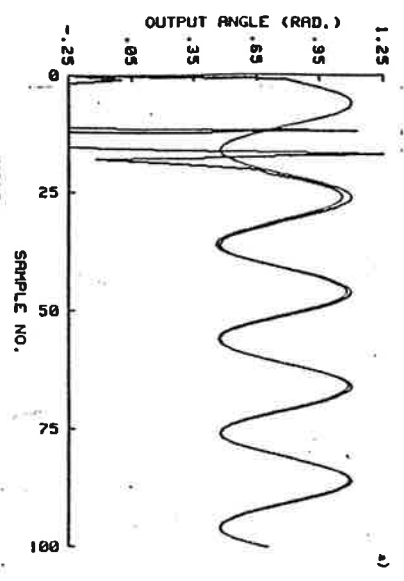
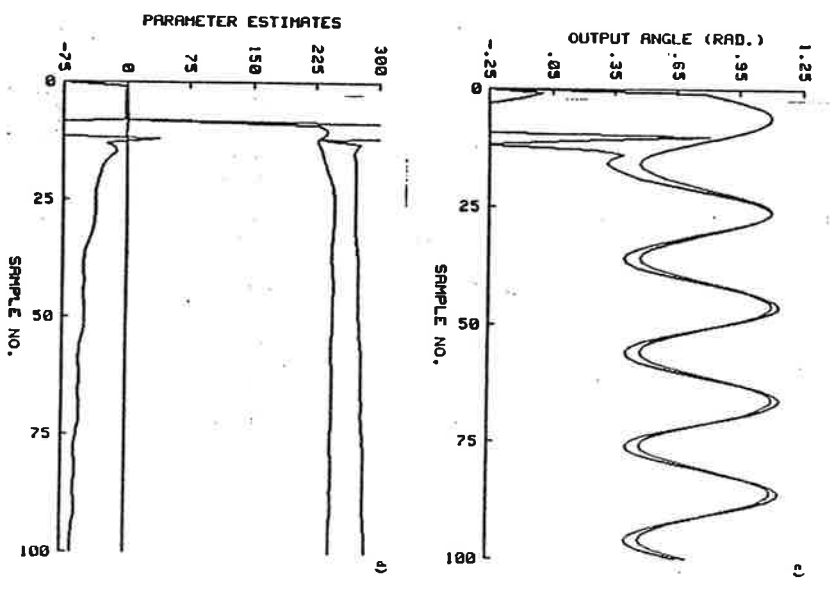
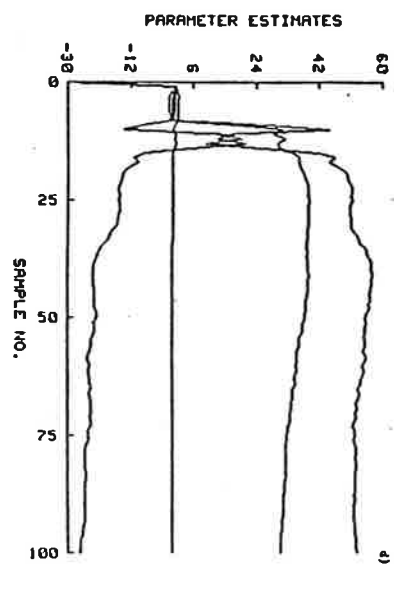
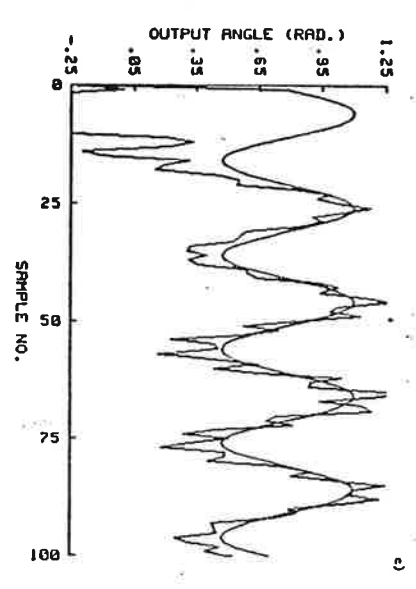


Figure 4. Response of nonlinear α), β) and linear c), d) when $w=2$, $l=2$.



III. CONTROL OF TWO LINKS

$$\begin{aligned}
 & \begin{bmatrix} d_{11} & & d_{12} \cos(\theta_{1k} - \theta_{2k}) \\ & d_{22} & \\ & & \end{bmatrix} A_{k+1} \\
 & + \begin{bmatrix} d_{13} & 0 \\ 0 & d_{23} \end{bmatrix} V_k + \begin{bmatrix} d_{14} & 0 \\ 0 & d_{24} \end{bmatrix} S_k \\
 & + \begin{bmatrix} d_{15} & 0 \\ 0 & d_{25} \end{bmatrix} Z_k = U_k
 \end{aligned}$$

$$\boxed{J(V_k) A_{k+1} + D_3 V_k + D_4 S_k + D_5 Z_k = U_k}$$

$$A_{k+1} = \begin{bmatrix} y_{1k+1} - 2y_{1k} + y_{1k-1} & & \\ y_{2k+1} - 2y_{2k} + y_{2k-1} & & \end{bmatrix} \times \frac{1}{\Delta t^2} = \begin{bmatrix} a_{1k+1} \\ a_{2k+1} \end{bmatrix}$$

$$V_k = \begin{bmatrix} y_{1k} - y_{1k-1} \\ y_{2k} - y_{2k-1} \end{bmatrix} \times \frac{1}{\Delta t} = \begin{bmatrix} v_{1k} \\ v_{2k} \end{bmatrix}$$

$$S_k = \begin{bmatrix} \sin y_{1k} \\ \sin y_{2k} \end{bmatrix} = \begin{bmatrix} s_{1k} \\ s_{2k} \end{bmatrix}$$

$$Z_k = \begin{bmatrix} \sin(y_{2k} - y_{1k}) (y_{2k} - y_{2k-1})^2 \\ \sin(y_{1k} - y_{1k-1}) (y_{1k} - y_{1k-1})^2 \end{bmatrix}$$

Can also be written as

$$\begin{aligned}
 \Phi_{1k}^T \Theta_{1k} &= U_k \\
 \Phi_{2k}^T \Theta_{2k} &= U_k
 \end{aligned}$$

$$\Theta_i^T = [d_{i1}, d_{i2}, \dots, d_{i5}]$$

$$\Phi_{ik} = [a_{i1k}, a_{i2k}, v_{1k}, s_{1k}, z_{1k}]$$

$$\begin{aligned}
 j=2 \quad \text{if } i=1 \\
 j=1 \quad \text{if } i=2
 \end{aligned}$$

Typical Linear Model

$$Y_{k+1} + A_1 Y_k + A_2 Y_{k-1} = B_1 U_k + B_2 U_{k-1}$$

In simplest case

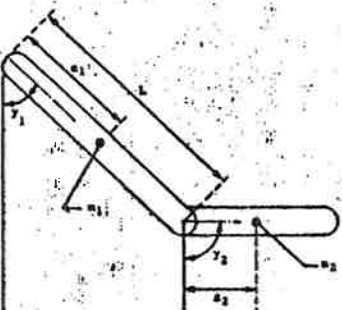
A_i, B_i diagonal

4 Parameters / row

General Case

2 Parameters / row

Equations for Two-Link Planar Manipulator



$$\begin{aligned}
 & \begin{bmatrix} (J_1 + m_2 l_2^2 + m_2 L_1^2) \ddot{\theta}_1 - m_2 L_1 l_2 \cos(\theta_2 - \theta_1) \ddot{\theta}_2 \\ m_2 L_1 l_2 \cos(\theta_2 - \theta_1) \ddot{\theta}_1 + (J_2 + m_2 l_2^2) \ddot{\theta}_2 \end{bmatrix} \begin{bmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \end{bmatrix} + \begin{bmatrix} K_1 & f_1 \\ K_2 & f_2 \end{bmatrix} \begin{bmatrix} \dot{\theta}_1 \\ \dot{\theta}_2 \end{bmatrix} \\
 & + \underbrace{\begin{bmatrix} (m_1 l_1 + m_2 L_1) g \sin \theta_1 \\ m_2 g l_2 \sin \theta_2 \end{bmatrix}}_{\text{Gravity}} + \underbrace{\begin{bmatrix} m_2 L_1 l_2 \sin(\theta_2 - \theta_1) \dot{\theta}_1^2 \\ m_2 L_1 l_2 \sin(\theta_2 - \theta_1) \dot{\theta}_2^2 \end{bmatrix}}_{\text{Centrifugal}} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}
 \end{aligned}$$

MODEL MATCHING CONTROL

Desired closed loop system

$$Y_{k+1} + E_1 Y_k + E_2 Y_{k-1} = H_1 R_k + H_2 R_{k-1}$$

For $E_1 = E_2 = H_2 = 0$ $H_1 = I$ $R_k = Y_{k+1}$
get Deadbeat

Rewrite Process Model ($A_{k+1} = Y_{k+1} - 2Y_k + Y_k$)

$$Y_{k+1} = T^2 J^{-1}(Y_k) [U_k - D_3 Y_k - D_4 S_k - D_5 Z_k + \frac{1}{\Delta t^2} J(Y_k) (2Y_k - Y_{k-1})]$$

$$U_k = D_3 Y_k + D_4 S_k + D_5 Z_k$$

$$+ \frac{1}{\Delta t^2} J(Y_k) [(Y_{k+1} - 2Y_k) - E_1 Y_k - E_2 Y_{k-1}] + H_1 R_k + H_2 R_{k-1}$$

Adaptive Implementation

1) Estimate θ_{ik} using the errors

$$e_{ik} = \phi_{ik}^T \theta_{ik} - u_{ik} \quad (\sigma = \phi_{ik}^T \theta_i - u_{ik})$$

and two parallel sequential least squares estimators (one for θ_{1k} one for θ_{2k})

Two estimation problems of size 5 (vs 4 for linear)

4x3

2) Use θ_{ik} to generate

$$J_k(y_k) = \text{Est}(J(y_k))$$

$$D_{ik} = \text{Est}(D_i) \quad i=3,4,5$$

3) Use

$$u_k = D_{3k} v_k + D_{4k} s_k + D_{5k} z_k$$

$$+ \frac{1}{T} J_k(y_k) [y_{k+1} - z_k y_{k+2} - E_1 y_k - E_2 y_{k+1} + H_1 R_k + H_2 R_{k+1}]$$

Note
No division of time varying quantities

V. SIMULATION

Model

$$y_{k+1} - y_k + 0.29 y_{k-1} = 0.29 R_k$$

$R_k = \text{d.c. offset} + \text{sinusoids}$

$$T = 0.02$$

Link lengths = 1 foot

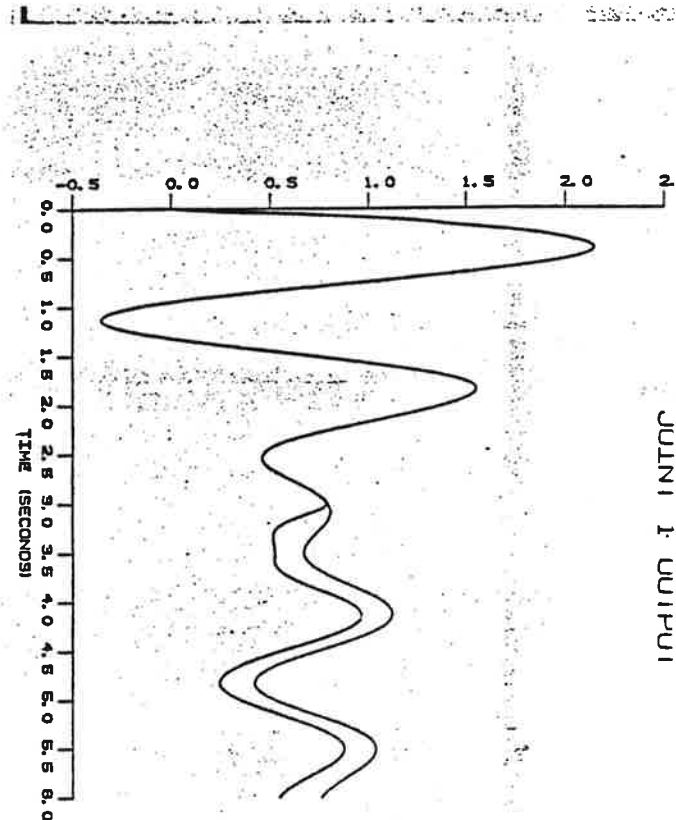
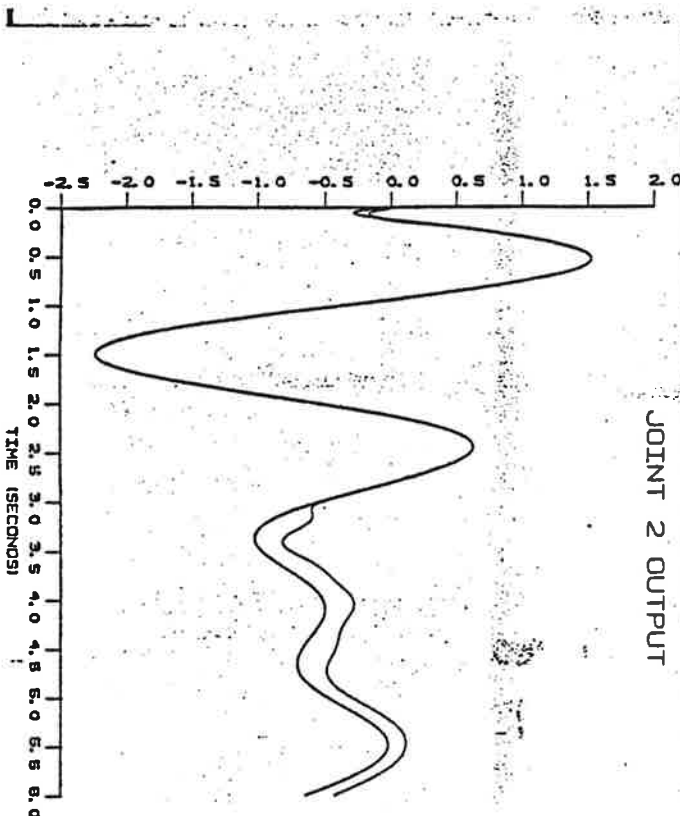
Masses = 1 lb mass $\omega_2 = 10 \text{ rad/sec}$

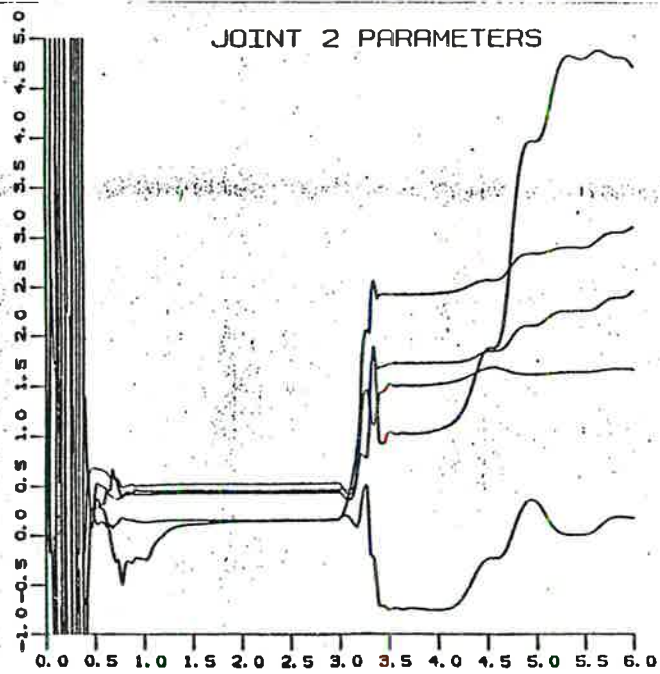
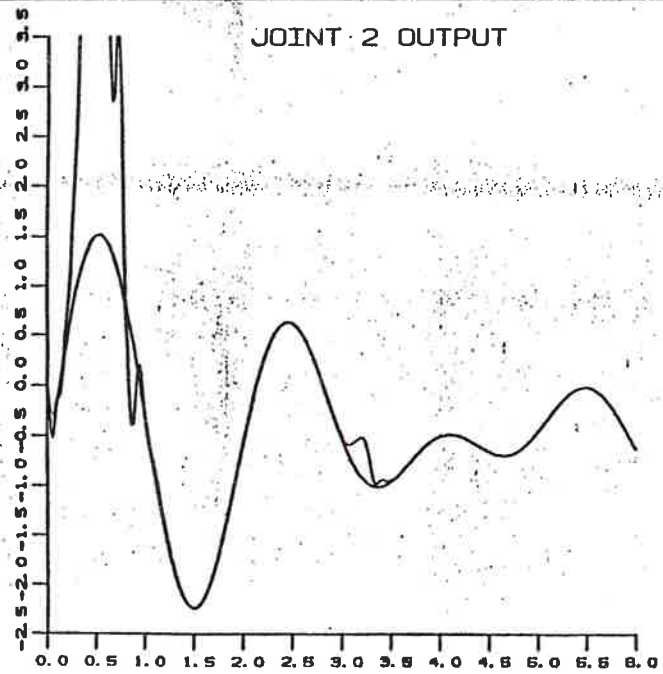
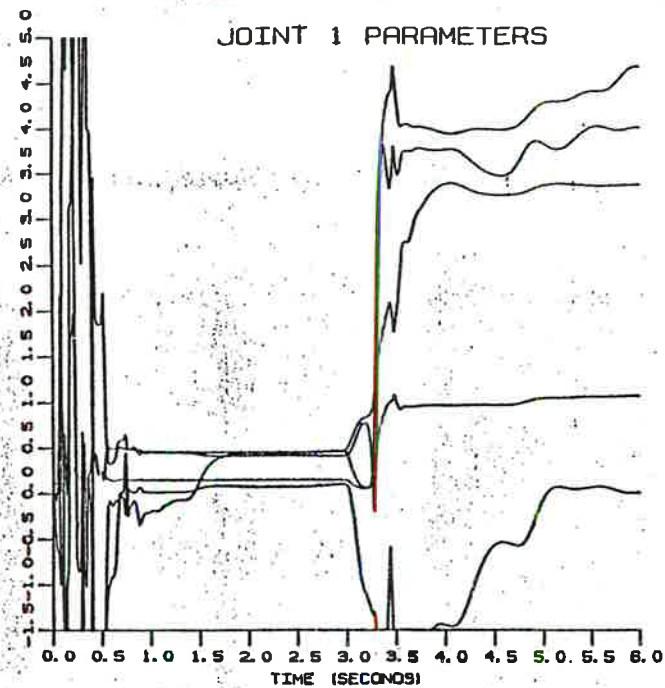
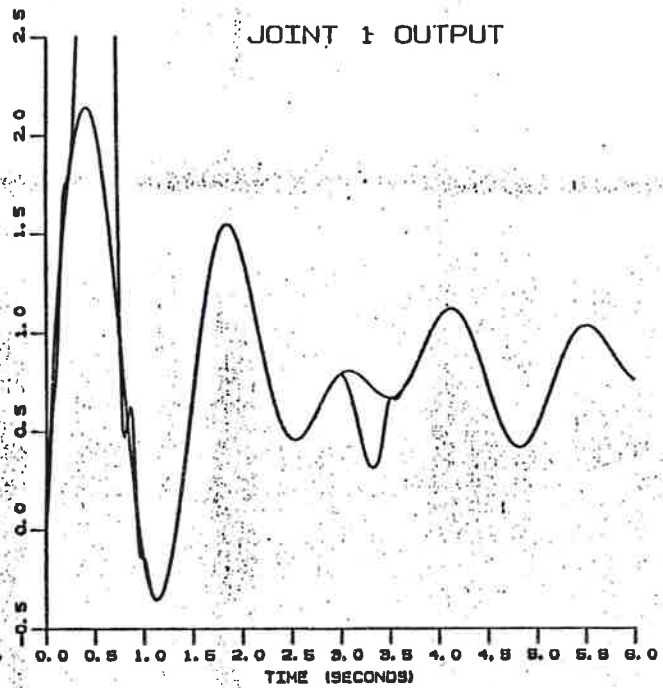
Damping = 1.5 ft-lb-sec

$$\lambda = 0.95$$

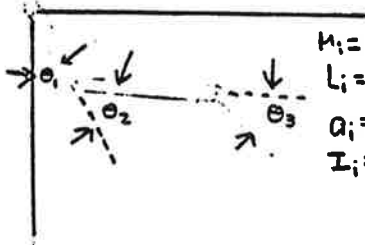
$$\theta_{10} = [1 \ 0 \ 0 \ 0]$$

$$\theta_{20} = [0 \ 1 \ 0 \ 0]$$





I. THREE LINK CASE:



M_i = mass link i
 L_i = length link i
 a_i = dist. to c.g. link i
 I_i = mom. Inertia link i

Eq 1

$$\begin{aligned}
 D_{11} \ddot{\theta}_1 + D_{12} \ddot{\theta}_2 + D_{13} \ddot{\theta}_3 + D_{14} [c\theta_2 (2\ddot{\theta}_1 + \ddot{\theta}_2) - s\theta_2 (\dot{\theta}_1^2 + 2\dot{\theta}_1 \dot{\theta}_2)] \\
 + D_{15} [c(\theta_2 + \theta_3) (2\ddot{\theta}_1 + \ddot{\theta}_2) - s(\theta_2 + \theta_3) (\dot{\theta}_1^2 + \dot{\theta}_2^2 + 2(\dot{\theta}_1 \dot{\theta}_2 + \dot{\theta}_1 \dot{\theta}_3 + \dot{\theta}_2 \dot{\theta}_3))] \\
 + D_{16} [c\theta_2 (2\ddot{\theta}_1 + 2\ddot{\theta}_2 + \ddot{\theta}_3) - s\theta_2 (\dot{\theta}_1^2 + 2\dot{\theta}_1 \dot{\theta}_2 + 2\dot{\theta}_1 \dot{\theta}_3)] \\
 + D_{17} \dot{\theta}_1 + D_{18} s\theta_1 + D_{19} s(\theta_1 + \theta_2) + D_{1,10} s(\theta_1 + \theta_2 + \theta_3) = T_1
 \end{aligned}$$

Eq 2

$$\begin{aligned}
 D_{21} [\ddot{\theta}_1 + \ddot{\theta}_2] + D_{22} \ddot{\theta}_3 + D_{23} [c\theta_2 (\ddot{\theta}_1) - s\theta_2 (\dot{\theta}_1^2)] \\
 + D_{24} [c(\theta_2 + \theta_3) (\ddot{\theta}_1) - s(\theta_2 + \theta_3) (\dot{\theta}_1^2)] \\
 + D_{25} [c\theta_2 (2\ddot{\theta}_1 + 2\ddot{\theta}_2 + \ddot{\theta}_3) - s\theta_2 (\dot{\theta}_1^2 + 2\dot{\theta}_1 \dot{\theta}_2 + 2\dot{\theta}_1 \dot{\theta}_3)] \\
 + D_{26} \dot{\theta}_2 + D_{27} s(\theta_1 + \theta_2) + D_{28} s(\theta_1 + \theta_2 + \theta_3) = T_2
 \end{aligned}$$

Eq 3

$$\begin{aligned}
 D_{31} [\ddot{\theta}_1 + \ddot{\theta}_2 + \ddot{\theta}_3] + D_{32} [c(\theta_1 + \theta_2) \ddot{\theta}_1 - s(\theta_1 + \theta_2) \dot{\theta}_1^2] \\
 + D_{33} [c\theta_2 (\ddot{\theta}_1 + \ddot{\theta}_2) - s\theta_2 (\dot{\theta}_1^2 - \dot{\theta}_2^2)] \\
 + D_{34} \dot{\theta}_3 + D_{35} s(\theta_1 + \theta_2 + \theta_3) = T_3
 \end{aligned}$$

Eq 1

$$\begin{aligned}
 D_{11} &= m_1 a_1^2 + m_2 (L_1^2 + a_2^2) + m_3 (L_1^2 + L_2^2 + L_3^2) + I_1 + I_2 + I_3 \\
 D_{12} &= m_2 a_2^2 + m_3 (L_2^2 + a_3^2) + I_2 + I_3 \\
 D_{13} &= m_3 a_3^2 + I_3 \\
 D_{14} &= m_2 L_1 a_2 + m_3 L_1 L_2 \\
 D_{15} &= m_3 L_1 a_3 \\
 D_{16} &= m_3 L_2 a_3 \\
 D_{17} &= K_1 \\
 D_{18} &= m_1 a_1 g + m_2 L_1 g + m_3 L_1 g \\
 D_{19} &= m_2 a_2 g + m_3 L_2 g = g D'_{19} \\
 D_{1,10} &= m_3 a_3 g
 \end{aligned}$$

Eq 2

$$\begin{aligned}
 D_{22} &= D_{13} \\
 D_{23} &= D_{14} = L_1 (m_2 a_2 + m_3 L_2) = L_1 D'_{19} \\
 D_{24} &= D_{15} \\
 D_{25} &= D_{16} \\
 D_{26} &= K_2 \\
 D_{27} &= g D'_{13} \\
 D_{28} &= D_{1,10}
 \end{aligned}$$

Eq 3

$$\begin{aligned}
 D_{31} &= D_{11} \\
 D_{32} &= D_{15} = L_1 (m_2 a_2) = L_1 D'_{19} \\
 D_{33} &= D_{16} = L_2 L'_{32} \\
 D_{34} &= K_3 \\
 D_{35} &= D_{1,10} = g D'_{19}
 \end{aligned}$$

NET RESULT

Estimate:

$$\Theta_1 = [D_{11}, D_{17}, D_{17}]$$

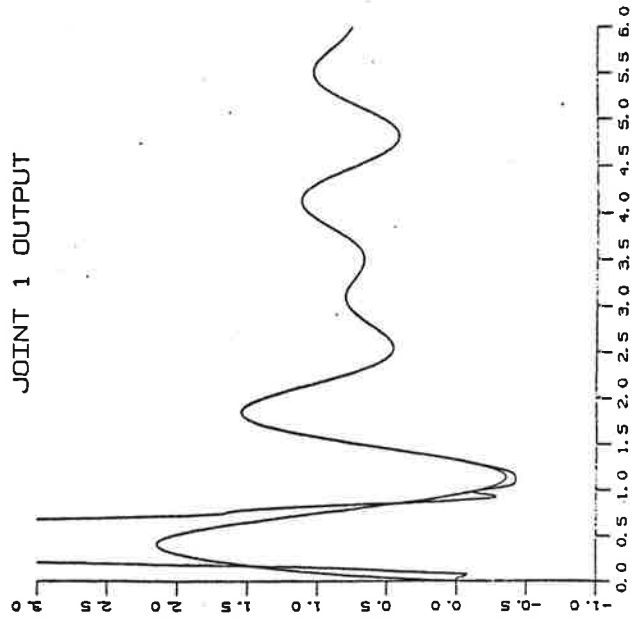
$$\Theta_2 = [D_{21}, D'_{23}, D_{26}]$$

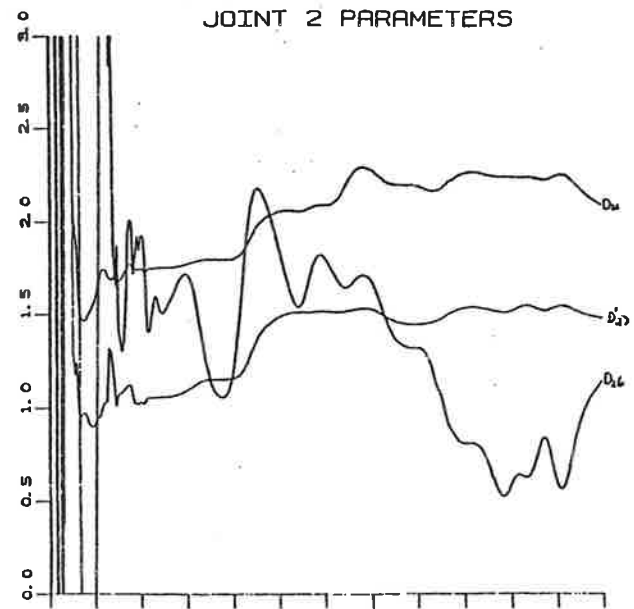
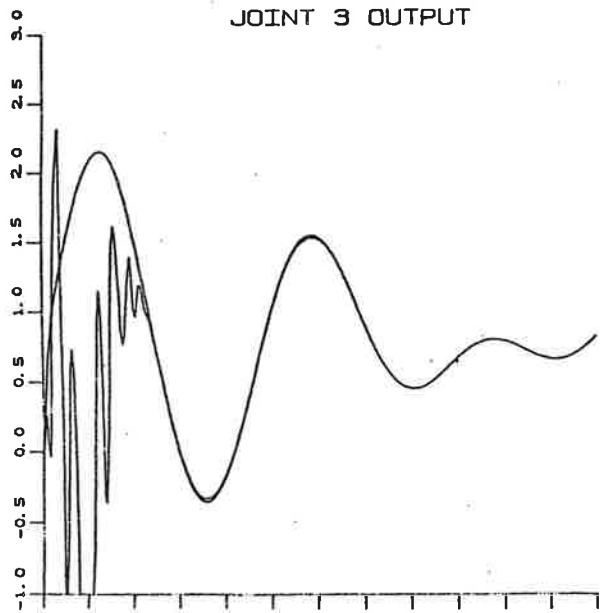
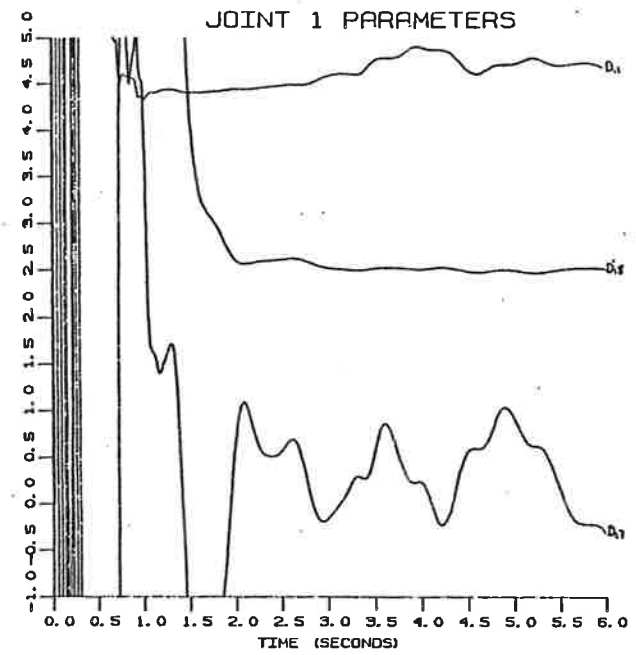
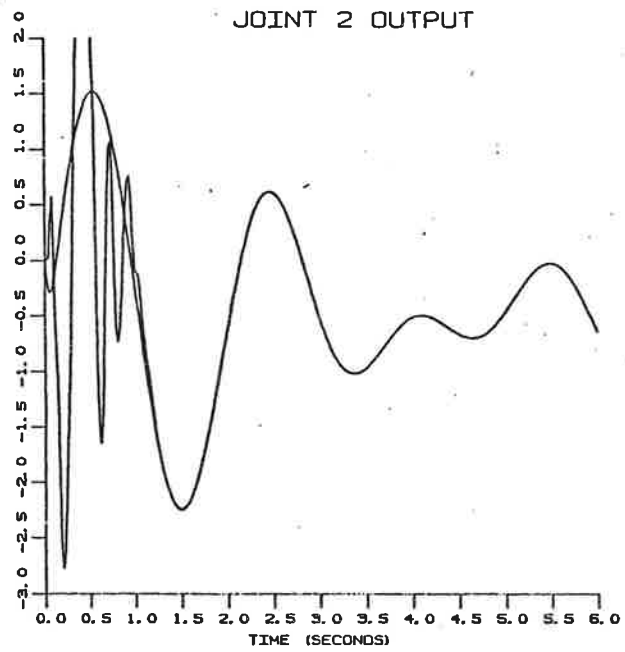
$$\Theta_3 = [D_{31}, D'_{32}, D_{34}]$$

$$X_i^T \Theta_i = Y_i$$

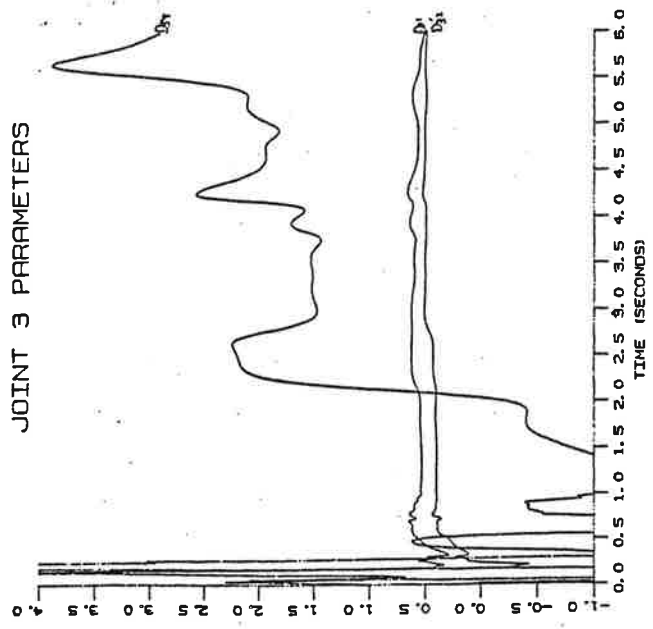
Measurable Signals

CONTROL STRATEGY
as before





JOINT 3 PARAMETERS



VI. DIGITAL CONTROLLER DESIGN USING EULER APPROX FOR SERVOS

Euler Approximation
 $T \ddot{u} + K \dot{u} = u$

$$\dot{u} \approx \frac{u(k) - u(k-1)}{T}$$

$$\ddot{u} \approx \frac{\dot{u}(k) - \dot{u}(k-1)}{T} = \frac{u(k) - 2u(k-1) + u(k-2)}{T^2}$$

Approx for stability $0 < T < 2$

① $(z+1)u = Xu$ as servos

↓ Euler Approx

$$u(k) + (z-2)u(k-1) + (1-K)u(k-2) = Xu(k)$$

② NO ZEROS

Will new design be stable?
 look at closed loop pole locations
 when Euler Based Controller Applied to Step-Inv. Model
 How does new design perform?
 comparatively?
 Simulate

Questions:

Pole Placement
 $AL+BP = An$
 $T = P$

Model Matching
 $AL+BP = AmB'$
 $T = Bm$
 $An = 1 + Amq^1 + Amq^2$
 $Bm = Bmq^1 + Bmq^2$
 $A = 1 + a_1q^1 + a_2q^2$
 $B = b_1q^1 + b_2q^2$
 $= b_1B'$

Final expression
 $U(z) = -T(z)U_c(z) + T(z)u$

ONE ZERO
 USUAL
 NEAR
 $z = -1$

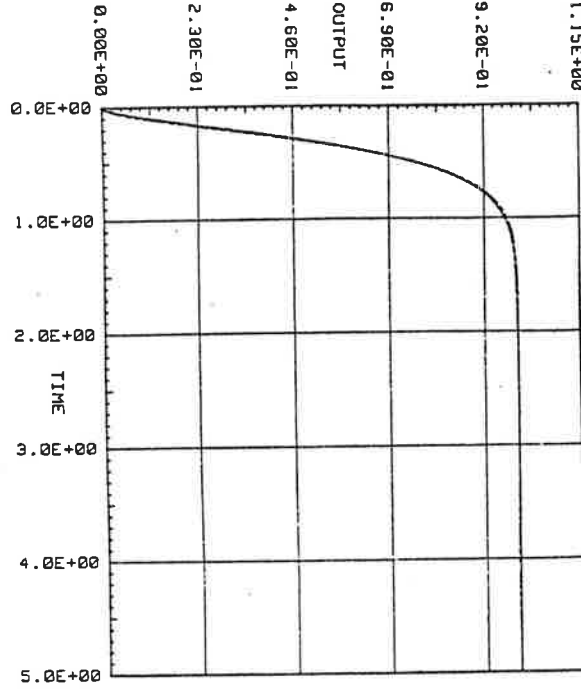
STEP INVARIANT APPROX

$$y + xz = Xu$$

Recall usual approach

MODEL MATCHING CONTROL BASED ON ③
 $u_k = \frac{1}{T} (xT-2 - Am) y_k + \frac{1}{T} (1-KT - Am) y_{k-1}$
 $+ \frac{1}{T} (Bmq^1 + Bmq^2)$
 $(y_{k+1} + Am y_k + Am y_{k-1} = Bmq^1 + Bmq^2)$
 POLE PLACEMENT USING ERROR FEEDBACK
 $u_k = \frac{1}{T} (xT-2 - Am) e_k + \frac{1}{T} (1-KT - Am) e_{k-1}$
 $e_k = r - y_k$

1.000	ROLLER GAIN: 1.000	MAGNITUDE: 1.000	PHASE: 0.000
1.002	ROLLER GAIN: 1.002	MAGNITUDE: 1.002	PHASE: 0.000
...
1.005	ROLLER GAIN: 1.005	MAGNITUDE: 1.005	PHASE: 0.000
...
1.010	ROLLER GAIN: 1.010	MAGNITUDE: 1.010	PHASE: 0.000
...
1.020	ROLLER GAIN: 1.020	MAGNITUDE: 1.020	PHASE: 0.000
...
1.030	ROLLER GAIN: 1.030	MAGNITUDE: 1.030	PHASE: 0.000
...
1.040	ROLLER GAIN: 1.040	MAGNITUDE: 1.040	PHASE: 0.000
...
1.050	ROLLER GAIN: 1.050	MAGNITUDE: 1.050	PHASE: 0.000
...
1.060	ROLLER GAIN: 1.060	MAGNITUDE: 1.060	PHASE: 0.000
...
1.070	ROLLER GAIN: 1.070	MAGNITUDE: 1.070	PHASE: 0.000
...
1.080	ROLLER GAIN: 1.080	MAGNITUDE: 1.080	PHASE: 0.000
...
1.090	ROLLER GAIN: 1.090	MAGNITUDE: 1.090	PHASE: 0.000
...
1.100	ROLLER GAIN: 1.100	MAGNITUDE: 1.100	PHASE: 0.000



DESIGNED MODEL POLES ARE:

MOTOR TIME CONSTANT = 5

1.000	ROLLER GAIN: 1.000	MAGNITUDE: 1.000	PHASE: 0.000
1.002	ROLLER GAIN: 1.002	MAGNITUDE: 1.002	PHASE: 0.000
...
1.010	ROLLER GAIN: 1.010	MAGNITUDE: 1.010	PHASE: 0.000
...
1.020	ROLLER GAIN: 1.020	MAGNITUDE: 1.020	PHASE: 0.000
...
1.030	ROLLER GAIN: 1.030	MAGNITUDE: 1.030	PHASE: 0.000
...
1.040	ROLLER GAIN: 1.040	MAGNITUDE: 1.040	PHASE: 0.000
...
1.050	ROLLER GAIN: 1.050	MAGNITUDE: 1.050	PHASE: 0.000
...
1.060	ROLLER GAIN: 1.060	MAGNITUDE: 1.060	PHASE: 0.000
...
1.070	ROLLER GAIN: 1.070	MAGNITUDE: 1.070	PHASE: 0.000
...
1.080	ROLLER GAIN: 1.080	MAGNITUDE: 1.080	PHASE: 0.000
...
1.090	ROLLER GAIN: 1.090	MAGNITUDE: 1.090	PHASE: 0.000
...
1.100	ROLLER GAIN: 1.100	MAGNITUDE: 1.100	PHASE: 0.000

CONTROL GAIN:

1.002	CONTROL GAIN: 1.002	MAGNITUDE: 1.002	PHASE: 0.000
1.010	CONTROL GAIN: 1.010	MAGNITUDE: 1.010	PHASE: 0.000
...
1.050	CONTROL GAIN: 1.050	MAGNITUDE: 1.050	PHASE: 0.000
...
1.100	CONTROL GAIN: 1.100	MAGNITUDE: 1.100	PHASE: 0.000

DESIGNED MODEL POLES ARE:

MOTOR TIME CONSTANT = 5

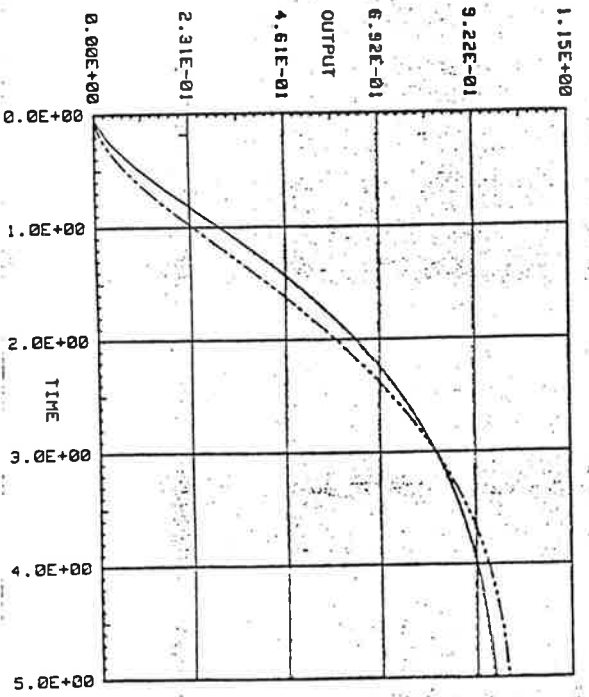
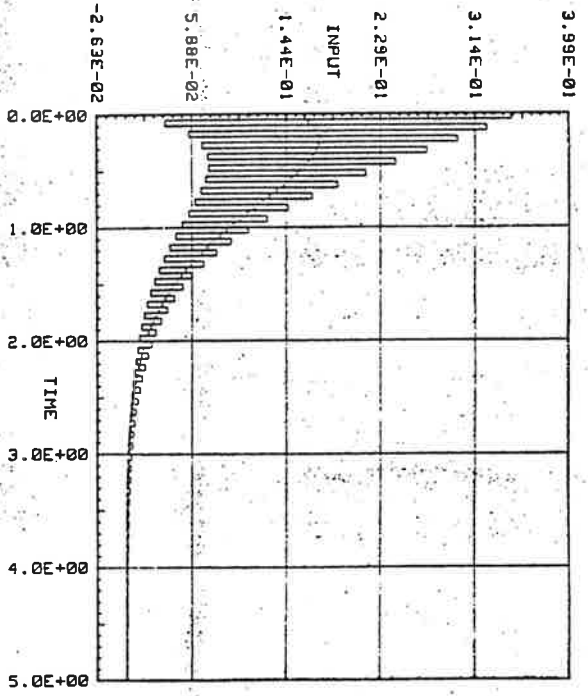
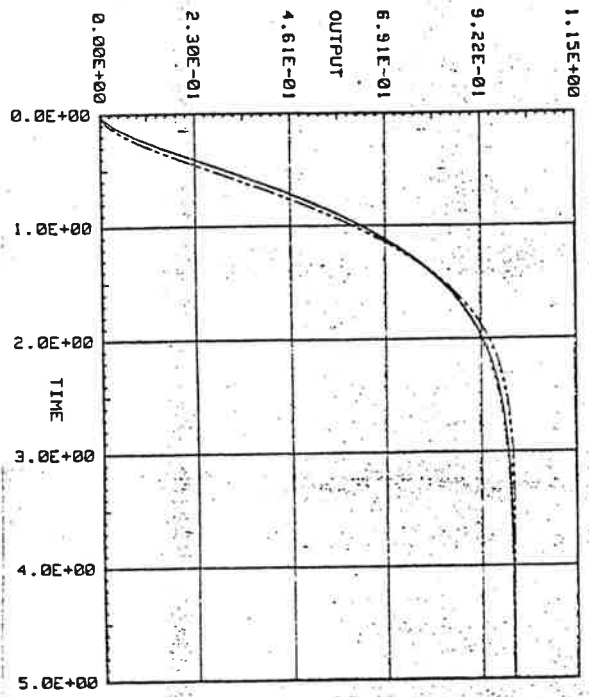
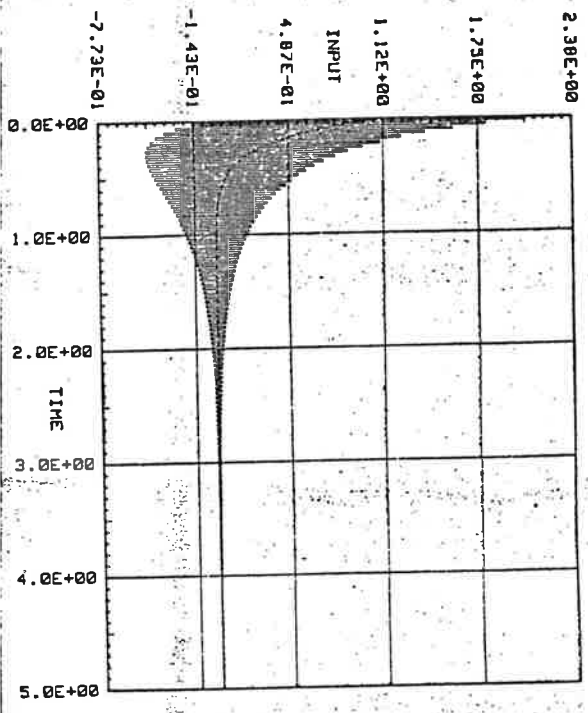
CONTROL GAIN:

ROLLER GAIN:

MAGNITUDE:

PHASE:

...



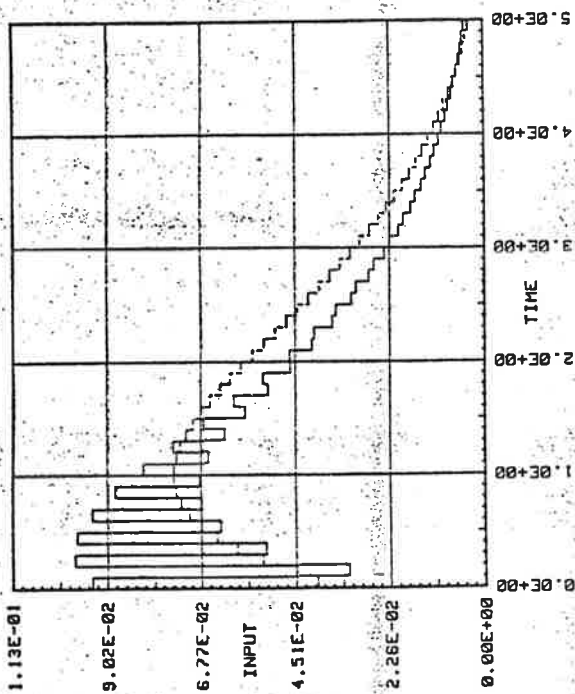
V. STABILITY ANALYSIS

Can we justify methodology with any Theory?

How good is it?

1. Can we apply control theory to fine adaptive control systems to show good stability in controlling unknown dynamic model?

2. When are it more easy to design adaptive control? Controller using better model or only using any?



THEOREM 1:

If the system to be controlled is the same time delay approx. model, then

the system is globally stable, i.e.

$$\|y_k\| \leq C$$

from the any int. conditions

$$\|y_k\| \leq C, \|u_k\| \leq C, \|e_k\| \leq C$$

or

$$\lim_{k \rightarrow \infty} \|y_k\| = 0$$

$$(1 + A_m q^k + A_m q^{2k}) \|y_k\| = (C_1 q^k + C_2 q^{2k})$$

Provided

$$|\det J_k(y_k)| > \epsilon, \forall k \geq 0$$

Proof: Complete for 1 DOF case.

only (one function)

Outline of proof:

Use analysis methodology of Lyapunov, Bode and Consist. must show

$$\|y_k\| \leq K_1 + K_2 \|y_0\|$$

e_k = plant error

Can show

$$(1 + A_m q^k + A_m q^{2k}) \|y_k\| = T \|y_0\|$$

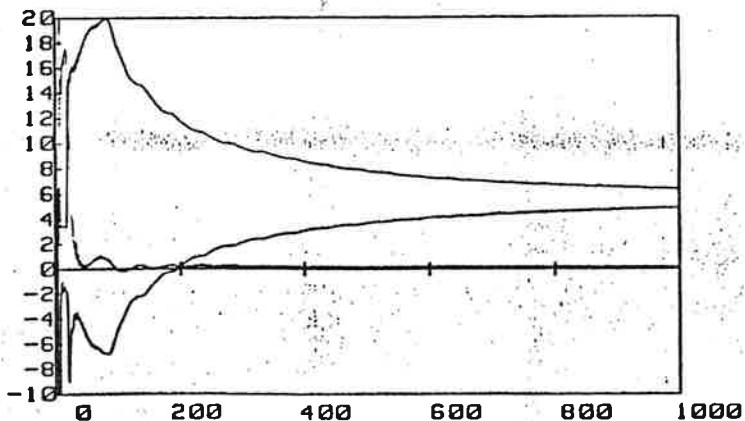
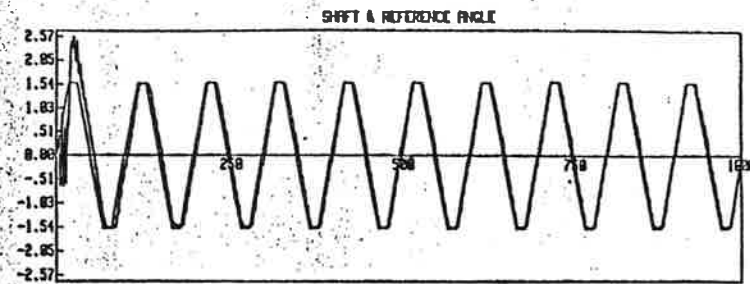
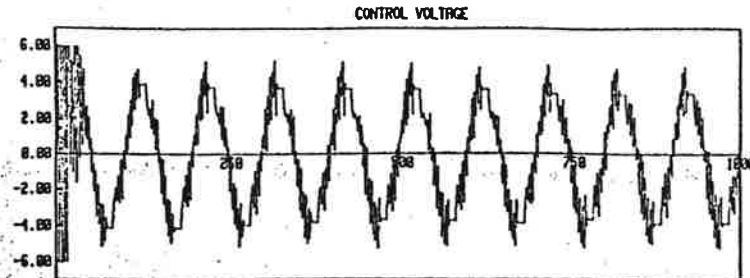
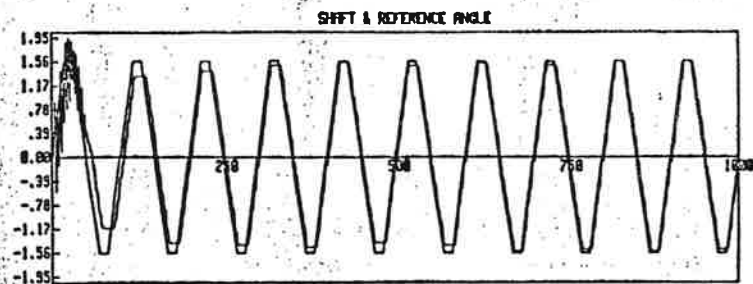
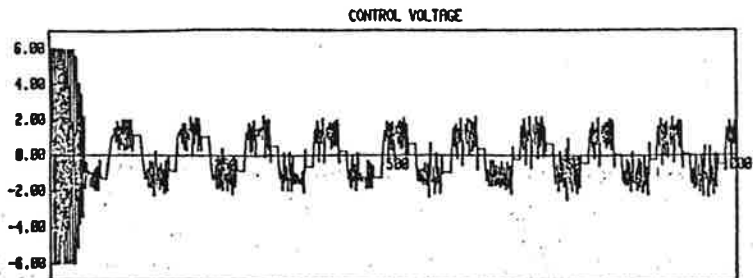
that is

$$\Rightarrow \|y_k\| \leq T \max_{k \geq 0} \|y_0\|$$

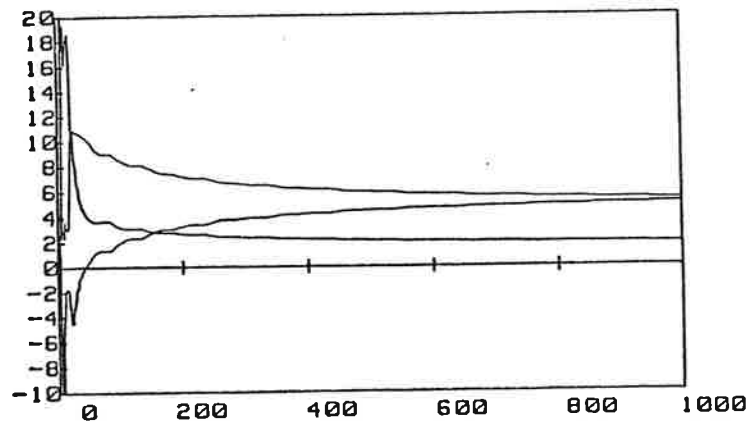
$$\textcircled{2} \|y_k\| = T \|y_0\| \text{ from } T \text{ is } T \text{ is } T$$

$$\Rightarrow \|y_k\| \leq T \max_{k \geq 0} \|y_0\|$$

③ Problem: I # of DOF ≥ 2 then $\phi_k \sim y_k^2$



PARAMETERS VS. SAMPLE NO.
 SAMPLE PERIOD = .1 SECONDS
 ___ A = 4.870E+00 ___ K = 6.311E+00
 ___ C/10 = 1.188E-01



PARAMETERS VS. SAMPLE NO.
 SAMPLE PERIOD = .1 SECONDS
 ___ A = 4.952E+00 ___ K = 5.224E+00
 ___ C/10 = 1.793E+00

Some desirable features of industrial adaptive controllers

Jan Sternby

Gambro AB
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Abstract

After a short introduction, three different physical processes will be described, for which adaptive control have been tried or considered. They are: a level control for a tank in a pulp production plant, an autopilot for ships, and a pressure and flow control system for a medical treatment system (artificial kidney). It will be discussed why adaptive control could be useful in these cases, and how it really works in two of them with certain algorithms.

Each process and its operators represent different demands on the adaptive control system such as robustness, variations in model structure, interpretability of identified parameters. These demands will be discussed for each process separately. This results in a small list of some desirable features of industrial adaptive controllers.

Some desirable features of industrial adaptive controllers

- Introduction
- Level control
- Autopilot
- Dialysis machine
- Summary

Algorithm:

- * Minimum Variance
- * Least Squares Est.
(b_0 fixed)
- * Feed forward (from chip feed)
- * $T_s = 1$ min.
- * $k = 3, 5, 6$
- * $\lambda = 0.96 - 0.995$

Level control

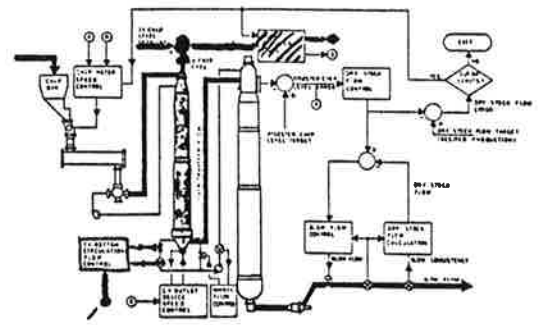


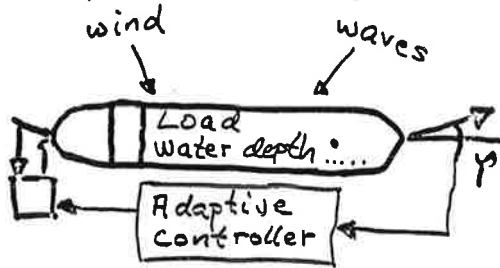
Fig. 1 - Principiell bild av ett vassakokeri.

- Simple level control with unknown actuator dynamics
- Time delay 2-3 min.
- On PID control, $T_s = 5$ sec., no feed forward
- Digital controller gives setpoint to analog controller

Experience:

- * PID structure ($NB=0$) robust, works well
- * Difficult to adjust k if $NB \neq 0$
- * Desirable to be able to compare parameters with PID control
- * Excess of parameters causes covariance blow-up (feed forward)

Autopilot



- * Adaptive controller gives setpoint for rudder machinery (nonlinear, partly unknown)
- * Integrator + Time Constant (time to go 1 ship length) + bias (wind...)
- * Varying noise conditions
- * Constant ship parameters

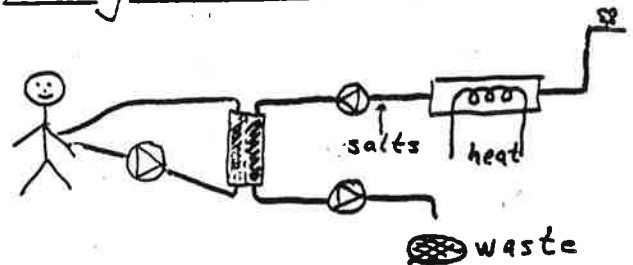
Experiences:

- * Integral action important (quick)
- * Natural criterion good as motivation
- * Essential to be able to handle different levels of noise
- * Quarterly sea difficult - should separate models for dynamics & disturbance
- * Would like to fix physical parameters

Algorithm:

- * Optimal LQ control
- * Spectral factorization
- * ELS estimation
- * Forced integrator
- * $\text{tr } P \leq \text{constant}$
- * $NA = 1$ (+ known integrator)
- * time delay $\leq NB$ (= 3)
- * $T_s = 5 \text{ sec.}$ (3 sec.)
- * $\lambda = 0.99 - 0.9999$

Dialysis machine



- * Control of:
 - Temperature
 - Conductivity
 - Flow
 - Pressure
- * Simple time constant 0.5 - 5 sec.
- * Dynamics vary with:
 - Aging of pumps (Gain)
 - Filter type (time const)
 - (Flow)
 - Blood composition(s)

Demands on adaptive control

- SAFETY!
No risk for sudden or slow loss of control accuracy.
→ Robustness
- Should identify filter = once/treatment
compensate for pump aging = long term drift
⇒ separate known models and unknown parts
Some physical parameters fixed
- PID-structure good for servability.

Some desirable features

* Safety

- * Avoid manual tuning of critical parameters (k..)
- * Robustness for model errors (e.g. rudder machinery)
- * PID-structure (NB=0) robust(?) and well-known
- * Automatic detection of parameter excess (Covariance blow-up)
- * Possibility to fix physical parameters
- * Separation dynamics - disturbance
- * Capability to handle varying noise levels

* MUST cope with non-minimum phase

* Include analog prefilter in self-tuner

→ * Feedforward essential

Self-tuning regulator with increased prediction horizon

Karl Johan Åström and Björn Wittenmark

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Lund Institute of Technology
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The basic self-tuning regulator based on least squares estimation and minimum variance control is not suited for systems with zeros outside the unit circle. In the presentation it is shown that the basic algorithm often can be used also for this type of systems by increasing the prediction horizon of the regulator.

Examples are given which show how the algorithm can be used for non-minimum phase systems. Possible convergence points are discussed and some new results concerning local convergence of the algorithm are presented.

The new insight into the properties of the algorithm explains why it can be applied to a wide range of practical problems provided that some design parameters are correctly chosen.

Reference

Wittenmark B., Åström K.J. (1984): Practical issues in the implementation of self-tuning control. *Automatica* 20, 595-605, 1984.

SELF-TUNING REGULATOR WITH INCREASED TIME HORIZON

- * INTRODUCTION
- * THE BASIC ALGORITHM
- * INCREASED PREDICTION HORIZON
- * ANALYSIS
- * SIMULATIONS
- * CONCLUSIONS

PROPERTIES

If convergence then

1. $r_y(z) = 0 \quad z = d_0, \dots, d_0 + \deg S^*$
 $r_{yu}(z) = 0 \quad z = d_0, \dots, d_0 + \deg R^*$
2. Sufficiently complex regulator
 \Rightarrow Minimum variance control

Convergence if

- * Minimum phase plant
- * $\frac{1}{C(z)} - \frac{1}{2}$ strictly positive real

Local stability if

$$C(z_i) > 0$$

for all z_i such that $B(z_i) = 0$

THE BASIC ALGORITHM

PROCESS

$$A^*(q^{-1})y(k) = B^*(q^{-1})u(k-d_0) + C^*(q^{-1})e(k)$$

ALGORITHM

$$LS: y(k+d_0) = R^*(q^{-1})u(k) + S^*(q^{-1})y(k)$$

$$MV: u(k) = -\frac{S^*}{R^*}y(k)$$

INCREASED PREDICTION HORIZON

Use $d > d_0$ in the estimation

What will happen?

- * Good thing to do in practice
- * Property 1 still valid but with d_0 replaced by d , Wittenmark (1973)
- * Can handle some non minimum phase processes

ANALYSIS

$$A(q) y(k) = B(q) u(k) + C(q) e(k)$$

$$d_0 = \deg A - \deg B$$

$$R(q) u(k) = -S(q) y(k)$$

Closed loop system

$$y(k) = \frac{CR}{AR+BS} e(k) = \frac{F}{q^\alpha} e(k)$$

MA process

Pole placement interpretation

Minimum variance ($B^+ = B$, $A_m = q^{d_0-1}$, $A_0 = C$)

$$R = BF$$

$$A + S = q^{d_0-1} C \Rightarrow y(k) = \frac{BF}{q^\alpha} e(k)$$

$$\deg R = n-1$$

$$\deg S = n-1$$

RESULTS

1. If convergence then there is a possible convergence point that corresponds to MA process of order $n-1$
2. Local convergence can be analyzed

INCREASED TIME HORIZON

Assume $d = n - \alpha + 1$

$$AR + BS = q^\alpha C$$

$$\Rightarrow y(k) = \frac{CR}{q^\alpha} e(k)$$

$$\deg R = n-1$$

$$\deg S = n-1$$

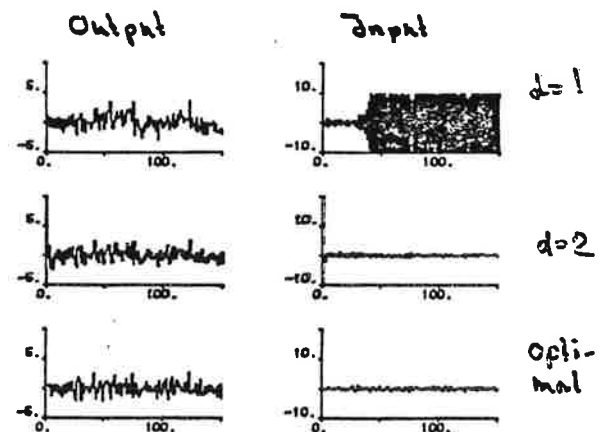
Comparison

- * Same degree of regulator
- * No cancellation of process zeros, compare Åström (1970)
- * Closed loop system is MA of order $\alpha = n-1$

SIMULATIONS

EXAMPLE 1

$$y(k) - y(k-1) = u(k-1) + 1.1 u(k-2) + e(k) - 0.5 e(k-1)$$



EXAMPLE 2

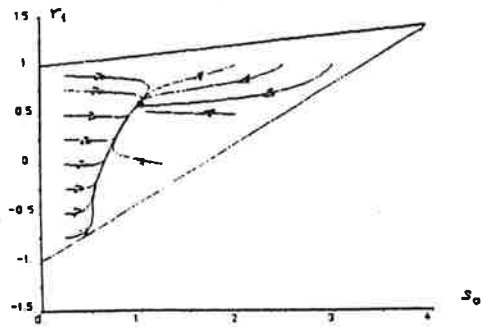
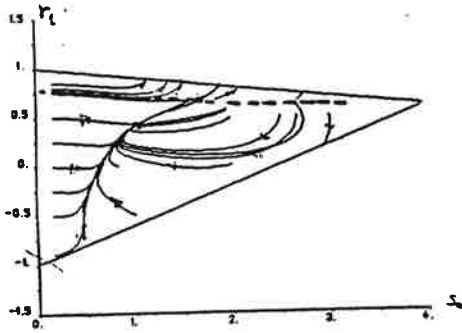
9

$$\dot{y} = u(t - \tau)$$

$$y(kh+h) = y(kh) + (h-\tau)u(kh) + \tau u(kh-h) + e(kh+h) + c(kh)$$

Non minimum phase if $\tau > h/2$

Simulation of ODE for the LS case



EXAMPLE 3

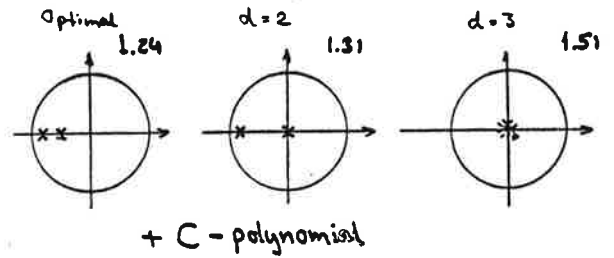
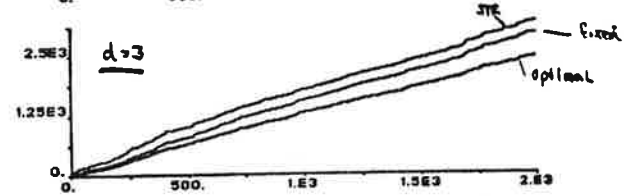
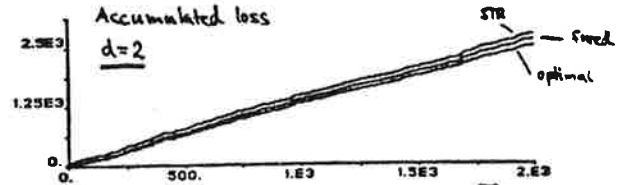
10

$$A(z) = z(z-1)(z-0.5)$$

$$B(z) = (z+2)(z+0.8)$$

$$C(z) = z^2(z-0.7)$$

$$u(k) = -\frac{s_0 + s_1 q^{-1}}{1 + r_1 q^{-1} + r_2 q^{-2}} y(k)$$



A universal control capable of stabilizing any single-input, single-output, minimum phase linear system of relative degree ≤ 2

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Within the past few years there have been developed several smooth dynamical controllers, not requiring "sufficiently rich" probing signals, which are capable of stabilizing any process which can be modelled by a linear system with transfer function of the form

$$T(s) = g \frac{\alpha(s)}{\beta(s)}$$

where g is a nonzero constant, and $\alpha(s)$ and $\beta(s)$ are monic, coprime polynomials, provided it can be assumed that

- 1/ $\alpha(s)$ is a strictly stable polynomial (i.e., Σ is minimum phase)
- 2/ a bound $n \geq \text{degree } \beta(s)$ is known
- 3/ the relative degree $n^* = \text{degree } \beta(s) - \text{degree } \alpha(s)$ is known exactly
- 4/ the sign of g is known

Since the preceding assumptions are very restrictive, there is ample motivation to see if controller structure can in some way be modified so that at least some of these assumptions can be avoided.

Prompted by this, we have just discovered that the 7-dimensional control system consisting of sensitivity function $\theta = [\theta_u, \theta_y]'$, where

$$\begin{aligned}\theta_u &= -\lambda\theta_u + u \\ \theta_y &= -\lambda\theta_y + y,\end{aligned}$$

filtered sensitivity function $\phi = [\phi_u, \phi_y, \theta_y]'$, where

$$\begin{aligned}\phi_u &= -\lambda\phi_u + \theta_u \\ \phi_y &= -\lambda\phi_y + \theta_y\end{aligned}$$

parameter adjustment law

$$k = \phi y,$$

Gain $N(x) = x \cos(x)$, where $x = k'k$, and feedback law

$$\begin{aligned}
u &= N(x)k'\theta + \left(\frac{\partial N(x)}{\partial x}\right)(k'\phi)^2 y + N(x)\phi'\phi y \\
&= x \cos(x)(k'\theta + \phi'\phi y) + (\cos(x) - x \sin(x))(k'\phi)^2 y \\
&= N(x)k'\theta + \phi' \frac{d}{dt}(N(x)k)
\end{aligned}$$

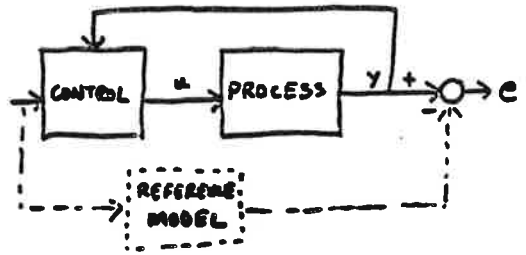
stabilizes any relative degree 2, minimum phase system of any dimension. What's surprising is that this control can also stabilize any relative degree 1 minimum phase system of any dimension. In other words, to achieve stability with the above control, it is only necessary to know that the minimum phase system to be controlled has relative degree not exceeding 2; i.e., it is not necessary to know relative degree exactly only an upper bound is required. It is natural to speculate that this should be true in general. In other words, with apriori knowledge of an upper bound \bar{n} , it should be possible to construct a smooth control system not incorporating a probing signal, which can stabilize any minimum phase system of any dimension, provided the system's relative degree does not exceed \bar{n} .

GENERAL ISSUES

Stability is of central importance - why?

What should be the role of a probing signal?

Should we insist on "smooth algorithms"



OBJECTIVES:

Tracking: $e \rightarrow 0$ as $t \rightarrow \infty$
 Internal Stabilization - all "states" bounded on $[0, \infty)$

Control Class

1. Smooth finite dimensional dynamical system
2. Arbitrary bounded reference input - "sufficiently rich" probing NOT incorporated for stabilization!

Process Model Assumptions - linear system with transfer function $g_p(d_p(s) / p_p(s))$

1. $d_p(s)$ stable - minimum phase
2. bound $n \geq \text{degree } p_p(s)$ known
3. rel. degree $n^* = \text{degree } p_p(s) - \text{degree } d_p(s)$ known
4. sign g_p known

2

3

Base Problem: Given $\dot{y} = ay + gu$, with a and g unknown and $g \neq 0$, does there exist an integer $m \geq 0$ and smooth functions $f: \mathbb{R}^{m+1} \rightarrow \mathbb{R}^m$, $h: \mathbb{R}^{m+1} \rightarrow \mathbb{R}$ such that the closed loop system

$$\begin{aligned} \dot{y} &= ay + g h(x, y) \\ \dot{x} &= f(x, y) \end{aligned}$$

is stable in the sense that for any initial state (x_0, y_0) , there exists a solution $(x(t), y(t))$ bounded on $[0, \infty)$ and $y(t) \rightarrow 0$ as $t \rightarrow \infty$?

If sign g known, the classical adaptive control with $n=1$, $f = y^2$, $h = -(\text{sign}(g))xy$ will stabilize:

$$\begin{aligned} \dot{y} &= (a - |g|x)y \\ \dot{x} &= y^2 \end{aligned}$$

The control $u = y^2 \sin(y)$ [suggested by Wonham] also stabilizes

$$\dot{y} = (a + g y \sin(y))y$$

but y doesn't necessarily go to zero due to multiple equilibria.

NEGATIVE RESULTS

Corollary If $m=1$ and if $f(x, y)$ and $h(x, y)$ are constrained to be quadratic polynomials in x and y , then stabilization of

$$\begin{aligned} \dot{y} &= ay + g h(x, y) \\ \dot{x} &= f(x, y) \end{aligned} \quad \} (1)$$

is impossible.

Lemma: If $m=1$ and if $f(x, y)$ and $h(x, y)$ are constrained to be rational functions in x and y , then stabilization of (1) is impossible.

Open Problem: Generalize the above by proving that Nussbaum's assertion is true for rational controllers of any dimension $m \geq 0$.

Nussbaum's Positive Result: It is possible to stabilize $\dot{y} = ay + gu$ with a smooth 1-dimensional controller.

Similar control $u = N(x)y$ $N(x) = x^2 \cos(x)$
 $\dot{x} = y^2$

$\dot{y} = (a + gx^2 \cos(x))y$ (1)
 $\dot{x} = y^2$ (2)

Analysis (Nussbaum)

$\dot{y}^2 = 2(a + gx^2 \cos(x))y^2$
 $\frac{dy^2}{dx} = 2(a + gx^2 \cos(x))$

$y^2(t) - y^2(0) = \int_{x(0)}^{x(t)} 2(a + gx^2 \cos(x)) dx$

$y^2(t) = \pi(x(t)) - \pi(x_0) + y^2(0)$

$\pi(\lambda) = 2a\lambda + 2g \int \lambda^2 \sin(\lambda) + 2\lambda \cos(\lambda) - \sin(\lambda)$
 ↑ dominates

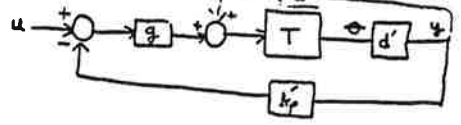
Let (A, b) be any $(n-1)$ -dimensional $\{(n-1)$ -dimensional $\}$, controllable pair with A stable. Define sensitivity function $\epsilon = [\epsilon_u, \epsilon_y, y]^T$ where

$\dot{\epsilon}_u = A\epsilon_u + bu$ $\epsilon \in \mathbb{R}^{2n-1} \Rightarrow \epsilon \in \mathbb{R}^{2\bar{n}-1}$
 $\dot{\epsilon}_y = A\epsilon_y + by$
 $y = d'\epsilon$

Lemma: For each $n_1 \leq n$ dimensional, minimum phase system of relative degree $n^* \leq \bar{n}$ and each monic, stable polynomial x^* of degree n^* there exist a nonzero constant g , a strictly proper, stable transfer matrix T , a strictly proper, stable transfer function L and a constant vector $k_p \in \mathbb{R}^{2n-1}$ [$k_p \in \mathbb{R}^{2\bar{n}-1}$] such that

$y = \frac{1}{x^*} (g(u - k_p^T \epsilon) + \epsilon + L(y))$
 $\epsilon = T(g(u - k_p^T \epsilon) + \epsilon + L(y))$

where ϵ is a linear combination of decaying exponentials.



Willems - Byrnes



$N(x) = x^2 \cos(x)$
 $\dot{x} = y^2$

SAME control just used to stabilize $\dot{y} = ay + gu$

parameterization: $p_p = (s-a)d_p + p$
 $p_p y = g d_p u$

$((s-a)d_p + p)y = g d_p u$
 $(s-a)y + p/d_p y = g u$

$\dot{y} = ay + gu + L(y)$ $L(y) = -\frac{p}{d_p} y$

Analysis: $\dot{y} = (a + gx^2 \cos(x))y + L(y)$
 $\dot{x} = y^2$

$\dot{y}^2 = 2(a + gx^2 \cos(x))y^2 + 2yL(y)$

$y^2(t) = \pi(x(t)) - \pi(x_0) + y^2(0) + 2 \int yL(y) dt$

$y^2(t) \leq \pi(x(t)) - \pi(x_0) + y^2(0) + c_1 + c_2 \int y^2 dt$
 $\leq c_1 + c_2 \int y^2 dt$
 $\leq c_1 + c_2 x$

$\pi(\lambda) = 2a\lambda + 2g \int \lambda^2 \sin(\lambda) + 2\lambda \cos(\lambda) - \sin(\lambda)$

DISCRETE-TIME ADAPTIVE CONTROL

FUNDAMENTAL PROBLEM

For the one-dimensional, discrete time system

$y(t+1) = ay(t) + gu(t)$

with a and g unknown and $g \neq 0$, does there exist an integer $m \geq 0$ and smooth functions

$f: \mathbb{R}^{m+1} \rightarrow \mathbb{R}^m$
 $h: \mathbb{R}^{m+1} \rightarrow \mathbb{R}$

such that the system

$y(t+1) = ay(t) + gh(x(t), y(t))$
 $x(t+1) = f(x(t), y(t))$

is stable in the sense that $(x(t), y(t))$ is bounded for $t \geq 0$ and $y(t) \rightarrow 0$ as $t \rightarrow \infty$.

Adaptive stabilization of linear multivariable systems

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Framework:

We have a system known only imprecisely, and we want to find a "universal stabilizing controller" which makes the output $y(t)$ of the system $\rightarrow 0$ as $t \rightarrow \infty$, while the parameters in the controller stay bounded.

I. The SISO case

Theorem. Consider the SISO system $\dot{x} = Ax + bu$; $y = cx$. Assume it is minimum phase and $cb > 0$. Define the controller $k = y^2$, $u = -ky$. Then, for all (x_0, k_0) it is true that $k_t \rightarrow k_\infty < \infty$ and $x_t \rightarrow 0$ as $t \rightarrow \infty$.

II. The MIMO case

Theorem. Consider the $m \times m$ system $\dot{x} = Ax + Bu$; $y = Cx$ and suppose that $\det G(s) = 0 \Rightarrow \text{Re}(s) < 0$ and that $\text{spec}(CB) \subset C^+$. Then, the controller $k = \|y\|^2$, $u = -ky$ satisfies $x_t \rightarrow 0$ and $k_t \rightarrow k_\infty$ as $t \rightarrow \infty$.

The proof consists of two ideas: First multivariable root-locus methods are employed to show that the eigenvalues will go into the left half plane, then "frozen analysis" as below shows that this is sufficient to deduce stability. An example of multivariable root locus was analyzed.

III. Frozen Eigenvalue Analysis

Theorem. Consider the system $\dot{x} = Ax - k(t)Bx$, where (i) for $k \gg 0$ $A - kB$ is stable, and, (ii) $k(t) \uparrow +\infty$ as $t \rightarrow \infty$. Then each solution $x(t)$ tends to 0 exponentially.

IV. Modifications and Extensions

The standard assumptions of necessary a priori knowledge for adaptive stabilization of an unknown SISO system are:

- 1) minimum phase
- 2) $n \leq N$, i. e. we have an upper bound on the order of the system
- 3) n^* , the relative degree of the system, is known
- 4) $\text{sign}(cA^{n^*-1}b)$ is known

We have shown that 2) is not needed, actually under weak condition the results can be generalized to a Hilbert space context. By a variant of Nussbaum's result, it was demonstrated that 4) is not needed either.

V. Necessary Conditions for Adaptive Stabilization

Metaprinciple. "Whatever can be done adaptively can be done if we know (A,b,c)."

Framework: Let the adaptive regulator be a n_z -dimensional linear system with internal state z , and dependent on a parameter k , which is updated according to $\dot{k} = f(k,y)$. By convergence we mean that $(x_t, z_t, k_t) \rightarrow (0, 0, k_\infty)$ as $t \rightarrow \infty$.

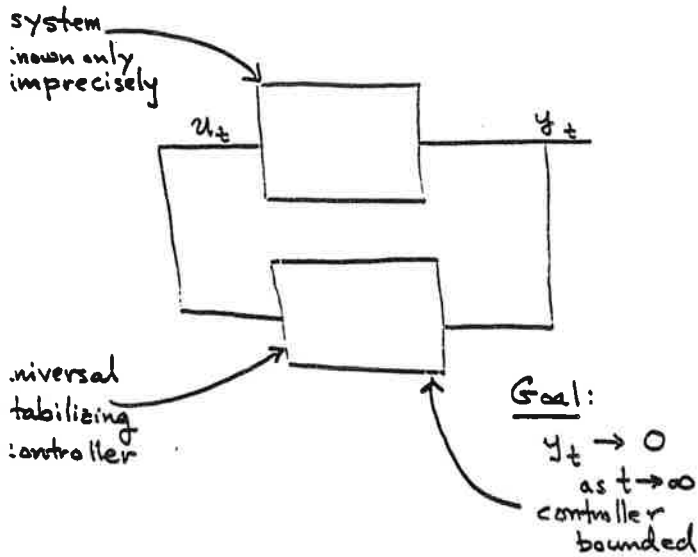
Theorem. If we have a convergent parameter adaptive stabilization scheme, then $n_z \geq n^* - 1$.

Theorem. Suppose $g(s) = c(sI - A)^{-1}b$ is minimum phase and of relative degree $\bar{n} \leq n^*$. Then, if $p(s)$ is minimum phase of degree n^* , the regulator

$$k(s) = \frac{k\beta^{n^*} p(s)}{(s + \beta)^{n^*}}, \quad \dot{k} = \gamma^2 k, \quad \beta = e^{-k}$$

will stabilize $g(s)$.

Adaptive Stabilization of Linear Multivariable Systems



Hypothesis : I. $g(s) = c(sI-A)^{-1}b = 0$

$\Rightarrow \text{Re}(s) < 0$

minimum phase

II. $g'(0) = cb > 0$

Theorem : Suppose

$\dot{x} = Ax + bu$

$y = cx$

(1)

is minimum phase with $cb > 0$. Define the controller

$k = y^2, u = -ky$ (2)

Then (1) - (2) satisfies $\forall (x_0, k_0)$:

(i) $k_t \rightarrow k_\infty$

(ii) $x_t \rightarrow 0$ as $t \rightarrow \infty$

Proof: $cb \neq 0 \Rightarrow b \notin \ker c$

L3

$\therefore \ker c + \text{span}\{b\} = \mathbb{R}^n$
 $(x_1, y) = x \in \mathbb{R}^n$

$\dot{x}_1 = A_{11}x_1 + A_{12}y$
 $\dot{y} = A_{21}x_1 + A_{22}y - \beta ky$
 $k = y^2$

1.3. $\text{sign}(\beta) = \text{sign}(cb) > 0, \text{spec}(A_{11}) = \mathbb{Z}(g(s)) \subset \mathbb{C}^-$

Pf. $\beta = cb$.

Proof 1: $g(s_0) = 0 \Rightarrow \hat{y}(s_0) = 0$
 where $x(0) = 0$

Therefore, at $s = s_0$, we have

$s_0 \hat{x}_1(s_0) = A_{11} \hat{x}_1(s_0)$

and $s_0 \in \text{spec}(A_{11})$.

Proof 2. $A_{11} = \overline{A+bf} : \mathcal{V}^* / \mathcal{R}^* \rightarrow \mathcal{V}^* / \mathcal{R}^*$
 where $\mathcal{V}^* = \ker c$
 $\mathcal{R}^* = (0)$

claim: It suffices to prove $|k_t| < M$,

because

- 1) k_t bounded $\Rightarrow \lim k_t = k_\infty$
- 2) A) $\Rightarrow y_t \in L^2(0, \infty)$
- 3) B) $\Rightarrow (x_1)_t \in L^2(0, \infty)$
- 4) A, B, C $\Rightarrow y \in L^2(0, \infty) \Rightarrow y_t \rightarrow 0$
- 5) A, B $\Rightarrow \dot{x}_1 \in L^2(0, \infty) \Rightarrow (x_1)_t \rightarrow 0$
- 6) D, E $\Rightarrow x_t = \begin{pmatrix} x_1 \\ y \end{pmatrix}_t \rightarrow 0$.

Lemma k_t is bounded

Proof. Set $V(x_1, y, k) = \frac{1}{2} y^2$.

$$\dot{V} = y A_{21} x_1 + \int A_{22} y^2 - \beta k y^2$$

$$V|_0^T = \int_0^T y A_{21} x_1 dt + A_{22} k \Big|_0^T - \beta k^2 \Big|_0^T$$

claim $|\int_0^T y A_{21} x_1 dt| \leq C_1 + C_2 \int_0^T y^2 dt$

$$\therefore -\frac{1}{2} y^2(0) \leq \alpha k \Big|_0^T - \beta k^2 \Big|_0^T + C_1$$

$$\therefore \gamma \leq \alpha k(T) - \beta k^2(T)$$

$$\therefore |k_t| \leq M.$$

II. Multivariable Linear Systems

$$\dot{x} = Ax + Bu, \quad y = Cx$$

(1) $G(s) = C(sI - A)^{-1}B$ m x m

Theorem (B-W) Suppose $\det G(s) = 0 \Rightarrow \text{Re}(s) < 0$ and $\text{spec}(CB) \subset \mathbb{C}^+$

Then, the controller

(2) $k = \|y\|^2, \quad u = -k y$

satisfies

$$x_2 \rightarrow 0, \quad k_t \rightarrow k_\infty.$$

claim: $|\int_0^T y A_{21} x_1 dt| \leq C_1 + C_2 \int_0^T y^2 dt$

set $z = A_{21} x_1, \quad z = A_{21} e^{A_{11}t} x_1(0) + L_2 y$
 $= L_1 x_1(0) + L_2 y$

L_1, L_2 bounded operators

$$\therefore |\langle y, z \rangle| \leq \|L_1 x_1(0)\| \|y\| + \|L_2 y\| \|y\|$$

$$\leq d_1 \|x_1(0)\| \|y\| + d_2 \|y\|^2$$

$$\leq d_1' \|x_1(0)\|^2 + d_2' \|y\|^2$$

$$= C_1 + C_2 \int_0^T y^2 dt$$

QED

Remarks 1. Multivariable Root-locus \Rightarrow "Frozen Anal."
 "Frozen Eigenvalue Analysis" \Rightarrow Stability

2. $\text{spec}(CB) = \mathbb{C}^-$

Multivariable Root-loci d'après Postlethwait
 Frozen Eigenvalue Analysis d'après -MacFarlane & Manton

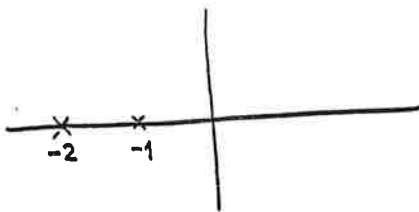
Then we ask 3 questions

- 1) Where do the root-loci start?
- 2) Where do the root-loci end?
- 3) How do they get there?

Example: $G(s) = \frac{1}{(1.25)(s+1)(s+2)} \begin{bmatrix} s-1 & s \\ -6 & s-2 \end{bmatrix}$

$K_k = \begin{bmatrix} k & 0 \\ 0 & k \end{bmatrix}, u = -k y$

- 1) $s_i(k) \in \text{Poles } G(s) = \{-1, -2\}$
- 2) $\lim_{k \rightarrow \infty} s_i(k) = \infty$, Zeros $G = \{0\}$
- 3) How do they get there?



- N.B. 1. $0 \leq k < 1.25$ stable
 2. $1.25 \leq k \leq 2.5$ unstable
 3. $k > 2.5$ stable

In particular $k \rightarrow s_i(k)$ is not 1-1!
 $m = p = 1 \quad 1 + k g(s) = 0 \Rightarrow k = -1/g(s)$

Indeed $\det(I + k G(s)) = 0$

Set $g = -1/k$

(*) $\det(gI - G(s)) = 0$

in our example

$0 = f(s, g) = 1.25[s^2 g^2 + 3s g^2 + 2g^2] - 2s g + 3g + \frac{4}{5}$

$g_{\pm}(s) = \frac{(2s-3) \pm \sqrt{1-24s}}{(2.5)(s+1)(s+2)}$

the algebraic functions have a branch point at

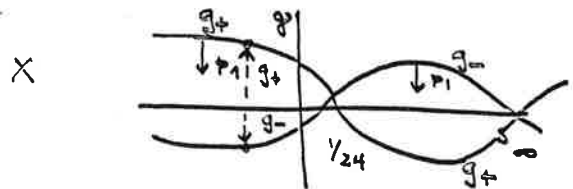
$s = 1/24$

Similarly, one computes

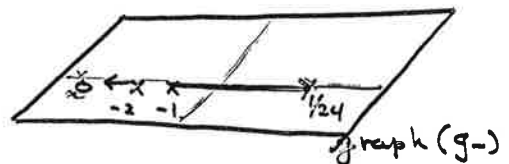
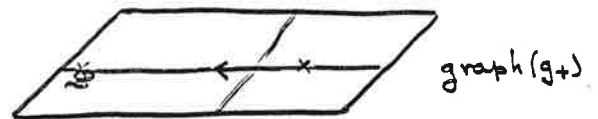
$s_{\pm}(g) = \frac{(2-3g) \pm \sqrt{9g-g^2}}{(2.5g)}$

$g_+(-2) = \infty, g_-(-1) = \infty \quad g_+(-2) = \frac{-3 + \sqrt{49}}{2.5}$

Consider $f(s, g) = 0$. This defines the algebraic curve / Riemann surface X



$X \subset S^2 \times S^2, \pi = \text{proj}_1(S, g) = s$



Newton (1680)

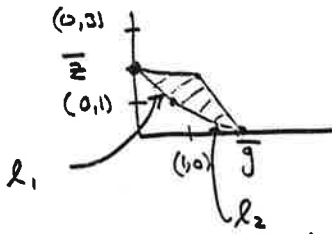
$$(1+x)^a = \sum a_i x^{(p/q)^i}$$

$$y = (1+x)^a : f(x,y) = y(1+x)^{-a} - 1 = 0$$

$$f(s,y) = c[s^2 y^2 + 3s y^2 + 2y^2] - 2s y + 3y + d = 0$$

Set $z = 1/s$ obtaining
 $f(z,y) = 0$

low does $z_i(y) \rightarrow 0$ as $y \rightarrow 0$



slope $l_1 = -a/1$, height = r

Newton: $\exists n$ branches of $z_i(y)$

$$z_i(y) = \alpha_i y^{1/q_i} + \dots$$

Scalar $\mathbb{I}/0$ l_1 has slope $-a/n$
 $z_i(y) = \alpha_i y^{1/n}$ ← Butterworth pattern

Frozen Eigenvalue Analysis

Theorem Consider the system

$$\dot{x} = Ax - k(t) Bx \quad (*)$$

where

(i) for $k \gg 0$ $A - kB$ is stable

(ii) $k(t) \uparrow +\infty$

Each solution to (*) tends to zero exponentially.

Proof: $x \in \mathbb{R}^n$, $A^{n \times n}$, $B^{n \times m}$

rank $B = m = n$

$$\therefore B = B_1 C_1, \quad B_1^{n \times m}, \quad C_1^{m \times n}$$

$$\ker C_1 + \text{span } B_1 = \mathbb{R}^n$$

Consider $(x_1, y) = x \in \mathbb{R}^n$ and (*)

$$\dot{x}_1 = A_{11} x_1 + A_{12} y$$

$$\dot{y} = A_{21} x_1 + A_{22} y - k(t) \beta y$$

I. $\text{spec}(B) = \text{spec}(C_1 B_1)$ ✓

II. $s \in \text{spec}(A_{11}) \Leftrightarrow \det Q(sI - A)^{-1} B_1 = 0$

Newton's Theorem $f(s,y) = 0$ has n branches

$$f(s,y) = \prod_{i=1}^n (s - s_i(y))$$

At any branch point, each $s_i(y)$ can be developed into a Newton-Puiseux expansion

$$s_i(y) = \alpha_i y^{p_i/q_i} + \dots$$

The sum of the orders = n .

Applications:

- 1) Root Locus Plots (Dorf & Schweppes - MacFarlane)
- 2) General Position Lemmas:

Lemma Given $G(s)$, $\delta(G(s)) = n$. There exists

$u = -ky$ such that $G_k(s)$ has distinct poles.

3) Adaptive Control: $G(s) \text{ m x m } \delta(G)$ a. suppose $\det G(s) = 0 \Rightarrow s \in \mathbb{C}^+$, $\text{spec}(KB) \subset \mathbb{C}$

Then $u = -ky$, $k = \|y\|^2$ is self-tuning. That is

- (i) $k_t \rightarrow k_\infty$
- (ii) $x_t \rightarrow 0$.

Frozen eigenvalue hypothesis implies

I. $\text{spec}(\beta) \subset \mathbb{C}^+$

II. $\text{spec}(A_{11}) \subset \mathbb{C}^-$

Choose coordinates $z = T_1 x$ so that β is real

$$z = \begin{bmatrix} \beta_1 & \beta_2 & \dots & \beta_r \\ \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots \\ \dots & \dots & \dots & \dots \end{bmatrix}$$

Then,

$$\frac{d}{dt} \left(\frac{1}{2} \|z\|^2 \right) = \langle z, A_{21} z \rangle + \langle z, A_{22} z \rangle - k(t) \langle \beta, z \rangle$$

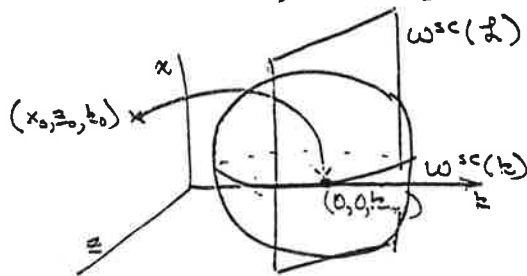
$$\leq x^T (A_{21}^T C_1 + C_1^T C_1 A^T) x - \beta \min k(t) \|z\|$$

$$\leq -c \|z\|^2 \quad t \gg 0.$$

$$\text{or } \|z_t\|^2 \leq e^{-ct} \|z_0\|^2 \quad t \gg 0$$

Theorem If we have a convergent parameter adaptive stabilization scheme, then $n_2 \geq n^* - 1$.

Proof



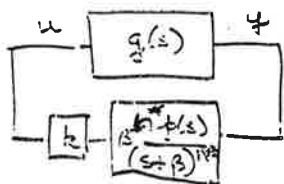
Lemma 1 $u = \dim W^u_{(0,0,k_0)} = 0$.

Pf Follows from Reduction Theorem of Shoshitaishvili / Palis-Takens :

$$\begin{aligned} \dot{x} &= f(x) \\ \dot{y} &= y \\ \dot{z} &= -z \end{aligned} \quad \text{local canonical form}$$

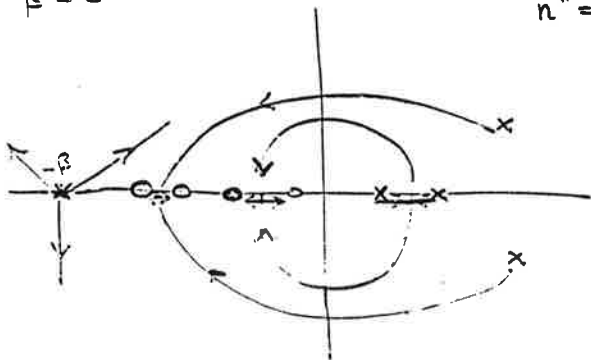
$x \in W^c, z \in W^s, y \in W^u$

$$J_{f_{(0,0,k_0)}} = \begin{bmatrix} A + bT_m c & bH_0 & 0 \\ G_0 c & T_m & 0 \\ k_x & 0 & k_z \end{bmatrix}$$



$$\begin{aligned} k &= \frac{c}{n^*} \\ \beta &= c \end{aligned}$$

$$n^* = 3$$



$$(s + \beta)^{n^*} d(s) + k \beta^{n^*} p(s) n(s) = 0$$

Frozen eigenvalue analysis: stable for $k \gg 0$
- algebraic function theory 'out'

Lemma 2 $\text{spec} \begin{pmatrix} A + bT_m c & bH_0 \\ G_0 c & T_m \end{pmatrix} = \overline{\sigma}^*$

Lemma 3 Consider $g(s) = \frac{1}{s^n - s^{n-1} - \dots - 1}$

suppose (F_0, G_0, H_0, J_0) stabilizes (internally) $g(s)$ and has degree n_2 . Then, $n_2 \geq n - 1$.

Q.E.D.

Theorem ($\mathbb{R} \neq \mathbb{C}$) Suppose $g(s)$:

$$\dot{x} = Ax + bu, \quad y = cx$$

is minimum phase and has relative degree n^*

If $p(s)$ is minimum phase of degree n^*

$$k(s) = \frac{k \beta^{n^*} p(s)}{(s + \beta)^{n^*}}$$

where $k = y^2, \beta = z$
stabilizes $g(s)$.

Sketch Assume $cA^{n^*-1}b > 0$

$$c(s) = k \frac{\beta^{n^*-1} p(s)}{(s + \beta)^{n^*-1}}$$

$p(s)$ Hurwitz of degree $n^* - 1$

$$g(s) = \frac{n(s)}{d(s)}$$

N.B. In closed loop, this is the same as

$$c'(s) = \frac{k \beta^{n^*-1}}{(s + \beta)^{n^*-1}}$$

compensating

$$g'(s) = \frac{p(s) n(s)}{d(s)}$$

$g'(s)$ has relative degree 1 !!
 $\ker_{\omega} c + \text{span}(b) = \mathbb{R}^n$
 $x_1 \quad y$

Modifications and Extensions

16

$$g(s) = c(I s - A)^{-1} b = \frac{p(s)}{q(s)}$$

1. minimum phase
 2. $n \leq N$ upper bound on order of system
 3. $n^* = \deg q - \deg p$ is known
 4. $\text{sign}(c A^{n^*-1} b)$ is known
- 2) is not needed, in fact $n = \infty$,

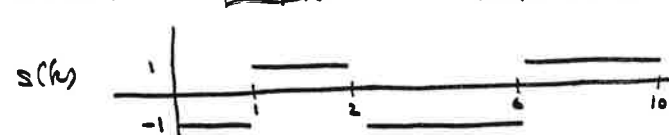
$$\begin{aligned} \dot{x} &= A x + b u & x &\in \mathcal{H} \\ y &= c x & u, y &\in \mathbb{R} \end{aligned}$$

$A: \mathcal{D}(A) \rightarrow \mathcal{H}$ densely defined
+ pure point spectrum.

- Lemmas
1. $A + b f^*$ pure point spectrum
- Weyl von-Neumann Theorem
 2. $\text{spec } A + b f^* = \mathcal{Z}(g(z))$
 $\Rightarrow A + b f^*$ stable: Hille-Yosida Thm.

examples delay or retarded systems, $u_t + u_{xxxx} = 0$
Flexible structures

Ex. $\dot{x} = ax + bu$, $\dot{z} = x^2$, $u = -\text{sign}(z) |z|$
Instead, set $u = \frac{s(z)}{k} |z|$ R. Nussbaum



so that Cesaro mean $C(s(k)k)$ satisfies
 $\lim_{k \rightarrow 1} C(s(k)k) = +\infty$ $\lim_{k \rightarrow 1} C(s(k)k) = -\infty$

$$\int_0^T \dot{v} dt = \frac{x^2}{2} \Big|_0^T = a \int_0^T x^2 + b \int_0^T s(z) |z| x^2 dt$$

$$= a k \Big|_0^T + b \int_0^T C(z) \Big|_0^T$$

$\therefore a k + b \int_0^T C(z)$ is bounded below
 $\therefore |k_{\pm 1}| \in M$ O.E.D.

Necessary Conditions in Adaptive Stabilization

"Whatever can be done adaptively can be done if we know (A, b, c) "

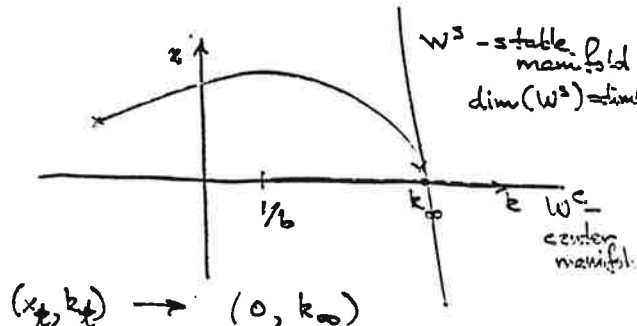


or,
Adaptive stabilization with smooth nonlinear controllers of $\dim \leq N$ implies (classical) stabilization with linear compensators of $\dim \leq N$.
(B. Morse)

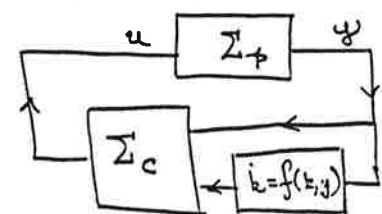
Proposition If \mathcal{C} is a collection of minimum phase systems which can be stabilized by some linear system of order $\leq N$, then for all $g \in \mathcal{C}$?
 $n^*(g) \leq N+1$.

Corollary For minimum phase systems; an upper bound on n^* is necessary for adaptive stabilization.

Ex $\dot{x} = x - k b x$
 $\dot{z} = x^2$



$$\begin{aligned} \dot{x} &= A x + b u, & y &= c x \\ \dot{k} &= f(k, y) \\ \dot{z} &= F(k) z + G(k) y \\ u &= H(k) z + J(k) y \end{aligned}$$



$(x_t, z_t, k_t) \rightarrow (0, 0, k_0)$ - Convergence

$$\dot{x}_1 = A_{11} x_1 + A_{12} y$$

$$\dot{y} = A_{21} x_1 + A_{22} y - k e^{r_k t} \frac{A_{21}}{A_{11}} x_1$$

$$\dot{w} = \begin{bmatrix} 0 & 1 & 0 \\ a & \dots & 0 \\ 0 & \dots & 0 \end{bmatrix} w + \begin{bmatrix} 0 \\ \dots \\ 0 \end{bmatrix}$$

$$\dot{y} = y^2$$

N.B. $\dot{x}_1 = A_{11} x_1$ is stable.

$$w = \begin{pmatrix} e^{-\dots t} \\ \dots \\ -e^{-\dots t} \end{pmatrix} w + \begin{pmatrix} 0 & 1 & 0 \\ a & \dots & 0 \\ 0 & \dots & 0 \end{pmatrix} w$$

are stable: minimum phase

$e^{k t} \uparrow + \infty$ + "frequency analysis"

$$\frac{d \|y\|^2}{dt} = A_{21} x_1 y + A_{22} y^2 - k e^{r_k t} \frac{A_{21}}{A_{11}} x_1 y$$

$$\approx c_1 + i_2 k + c_2 k - c_1 k^2$$

$\Rightarrow k$ bounded.

$\Rightarrow x_{t_1}, y_t \rightarrow 0 \quad k_t \rightarrow k_0$

Lyapunov functions, cost functions, and adaptive control

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Abstract

Lyapunov functions are derived for a class of discrete time adaptive systems. The derivations have been performed under the following assumptions: Reference value = 0, no non-minimum phase zeros of the control object, well damped desired closed loop poles. The parameter estimation is made via a gradient type of algorithm which makes use of the output error.

It is shown that the considered adaptive system have the following properties.

- Global stability in the sense of Lyapunov
- Exponential convergence to zero of the state vector and the output error
- Monotone convergence (not necessarily to zero) of the parameter errors

References

- Goodwin, G.C., Ramadge, P.J., Caines, P.E. (1980): Discrete Time Multivariable Control, IEEE-AC 25, 449-456.
- Hahn, W. (1967): Stability of Motion, Springer Verlag, New York, USA.
- Johansson, R. (1983): Multivariable Adaptive Control, Report TFRT-1024, Department of Automatic Control, Lund Institute of Technology, Lund, Sweden.
- LaSalle, J.P. (1976): The Stability of Dynamical Systems, SIAM, Philadelphia, USA.
- Mannerfelt, C.F. (1981): Robust Control Design with Simplified Models, Report TFRT-1021, Department of Automatic Control, Lund Institute of Technology, Lund, Sweden.
- Parks, P.C. (1966): Lyapunov Redesign of Model Reference Adaptive Control Systems, IEEE-AC 11, 362-367.

LYAPUNOV FUNCTIONS

COST FUNCTIONS

AND

ADAPTIVE CONTROL

*

ROLF JOHANSSON

LTH

- Problem
Stability of discrete-time
direct adaptive control
- Previous work
Lyapunov functions, SPR - Parks
BIBO - Goodwin-Ramadge-Caines
Egardt
- My approach
Deterministic case
State-space
Lyapunov function
- New results
Stability: Lyapunov-stability
Convergence: Exponential
(Global)

STABILITY CRITERIA

What conditions for stability prevail
in adaptive control theory?

- Strictly positive real transfer functions
- Persistency of excitation
- $\|\varphi\|$ limited

Criticism

- Implicit conditions
- No link between stability and
convergence

Remedy?

- Try to stick to "old" concepts like
 - Lyapunov functions
 - Cost criteria
 - Gain margin

CONTENTS

- Introduction
- Direct adaptive control
- What should a state-space
model contain?
- Parameter convergence
- Error and state dynamics
- Lyapunov function
- Stability
- Convergence
- Example
- Conclusions

DIRECT ADAPTIVE CONTROL

OBJECT, PLANT

$$y(t) = \frac{B_1^x(q^{-1})}{A^x(q^{-1})} u(t)$$

CONTROL LAW

$$R_u^x(q^{-1}) u(t) = -S_y^x(q^{-1}) y(t)$$

CLOSED-LOOP POLES ASSIGNMENT

$$R_u^x A^x + S_y^x B^x = T_1^x A_m^x B_r^x$$

PARAMETRIZATION

$$y = \frac{b_0 q^{-k} B_r^x}{T_1^x A_m^x B_r^x} [u(t) + \Theta^T \varphi(t)]$$

CONTROL LAW

$$u(t) = -\hat{\Theta}^T(t) \varphi(t)$$

PARAMETER ESTIMATION

$$\hat{\Theta}(t) = \hat{\Theta}(t-k) + \gamma(\varphi(t-k)) \varphi(t-k) e_f(t)$$

$$e_f(t) = T_1^x(q^{-1}) A_m^x(q^{-1}) e ; e = y$$

$$\gamma(\varphi(t)) = \frac{1}{\beta_0 \|\varphi(t)\|^2}$$

$$0 < \frac{\beta_0}{\beta_0} < 2$$

IDENTIFICATION DYNAMICS

$$\begin{cases} \hat{\Theta}(t) = \hat{\Theta}(t-k) + \gamma \varphi(t-k) e_f(t) \\ e_f(t) = b_0 q^{-k} [-\tilde{\Theta}^T(t) \varphi(t)] \end{cases}$$

$$\begin{aligned} \|\tilde{\Theta}(t)\|^2 - \|\tilde{\Theta}(t-k)\|^2 &= \\ &= -\alpha_e \frac{e_f^2(t)}{\|\varphi(t-k)\|^2} = \end{aligned}$$

$$= -\alpha_e \|\tilde{\Theta}(t-k)\|^2 \cos^2 \alpha(t-k)$$

α - Angle between φ and $\tilde{\Theta}$

! Notice that $\|\tilde{\Theta}(t)\|$ is determined already at time $(t-k)$! !

STATE-SPACE?

Dynamics

Control object x, y, u

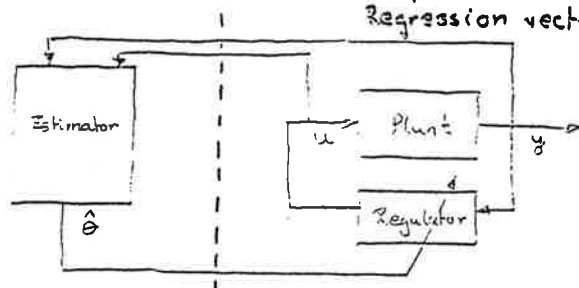
Regulator u, y

Estimator $\hat{\Theta}, \tilde{\Theta}$
 $\varphi, \varphi(t-k)$

$\tilde{\Theta}, \hat{\Theta}$ \longleftrightarrow $x, y, u, \varphi, \varphi(t-k)$

Parameters

Input
Outputs
Regression vectors



State at time t includes

$$\tilde{\Theta}(t+k-1), \dots, \tilde{\Theta}(t)$$

Introduce

$$z(t) \triangleq [\tilde{\Theta}^T(t+k-1) \dots \tilde{\Theta}^T(t)]^T$$

$$V_\Theta(z(t)) \triangleq \|z(t)\|^2 = z^T(t) z(t)$$

\Rightarrow

$$V_\Theta(z(t+1)) - V_\Theta(z(t)) =$$

$$= -\alpha_\beta \|\tilde{\Theta}(t)\|^2 \cos^2 \alpha(t)$$

α - angle between $\varphi, \tilde{\Theta}$

Standard proof of parameter convergence

ERROR DYNAMICS

More difficult?

Control object u, y, x

Regulator u, y

Regression vectors φ

IDEA #1

Relate all of them to $\xi(t)$!

$$u(t) = A^*(q^{-1}) \xi(t)$$

$$y(t) = B^*(q^{-1}) \xi(t)$$

$$x(t) = [x_1(t) \dots x_n(t)]^T$$

where $x_i(t) = \xi(t-i)$

$$\varphi(t) = [u(t-1) \dots u(t-k+1) y(t) \dots]^T$$

$$= M x(t)$$

How does $\xi(t)$ develop?

$$\xi(t) = \dots = \frac{1}{T_1^x A_m^x B_z^x} [-\tilde{\Theta}^T(t) \varphi(t)]$$

State-space formulation

$$x(t+1) = F x(t) + G (-\tilde{\Theta}^T(t) \varphi(t))$$

Remember that

$$\varphi = M x$$

State-space representation

$$x_i(t) \triangleq \xi(t-i)$$

$$\begin{bmatrix} x_1(t+1) \\ \vdots \\ x_n(t+1) \end{bmatrix} = \begin{bmatrix} -a_1 & \dots & -a_{n_A} \\ & \ddots & \\ & & 1 \end{bmatrix} \begin{bmatrix} x_1(t) \\ \vdots \\ x_{n_A}(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \\ \vdots \\ 0 \end{bmatrix} u(t)$$

$$y(t) = [0 \dots 0 b_0 b_1 \dots] x(t)$$

Controllable canonical form

Direct adaptive control reparametrization of $\xi(t)$!

$$\xi = \frac{R^x A^x + S^x B^x}{T_1^x A_m^x B_z^x} \xi =$$

$$= \frac{1}{T_1^x A_m^x B_z^x} [u + \Theta^T \varphi]$$

$$= \frac{1}{T_1^x A_m^x B_z^x} [-\tilde{\Theta}^T(t) \varphi(t)]$$

State-space formulation

$$\begin{bmatrix} x_1(t+1) \\ \vdots \\ x_n(t+1) \end{bmatrix} = \begin{bmatrix} -p_1 & \dots & -p_{n_p} & 0 & \dots & 0 \\ & \ddots & & & & \\ & & & 0 & & \\ & & & & & \\ & & & & & 1 \end{bmatrix} \begin{bmatrix} x_1(t) \\ \vdots \\ x_{n_p}(t) \\ \vdots \\ x_n(t) \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \\ \vdots \\ 0 \end{bmatrix} [-\tilde{\Theta}^T \varphi]$$

$$T_1^x A_m^x B_z^x = 1 + p_1 q^{-1} + \dots + p_{n_p} q^{-n_p}$$

$$x(t+1) = F x(t) + G (-\tilde{\Theta}^T(t) \varphi(t))$$

X

X

A STATE VECTOR

Parameter errors

$$z(t) = [\tilde{\Theta}^T(t+k-1) \dots \tilde{\Theta}^T(t)]^T$$

Error dynamics, regulator etc

$$x(t) = [\xi(t-1) \dots \xi(t-n)]^T$$

$$\underline{X}(t) = [z^T(t) \ x^T(t)]^T$$

$$\underline{X}(t+1) = \begin{bmatrix} 0 & \dots & 0 & 1 & 0 \\ 1 & 0 & \dots & 0 & 0 \\ 0 & \dots & 0 & 0 & 0 \\ \vdots & & & & \vdots \\ 0 & \dots & 0 & 0 & F \end{bmatrix} \underline{X}(t) + \begin{bmatrix} L(\underline{X}(t)) \\ 0 \\ \vdots \\ 0 \\ G \end{bmatrix} \nu(\underline{X}(t))$$

$$L[\underline{X}(t)] = \gamma(\varphi(t)) b_0 \varphi(t)$$

$$\nu[\underline{X}(t)] = -\tilde{\Theta}^T(t) \varphi(t)$$

$$V_x(x(t)) = \ln[1 + \|Qx(t)\|^2]$$

$$\begin{aligned} V_x(x(t+1)) - V_x(x(t)) &\leq \\ &\leq \alpha_x \|\tilde{\Theta}(t)\|^2 \cos^2 \alpha(t) \end{aligned}$$

α - angle between $\tilde{\Theta}$ and φ

Matching with

$$V_{\Theta}(z(t)) = \sum_{i=0}^{k-1} \|\tilde{\Theta}(t+i)\|^2$$

gives

$$V[\underline{X}(t)] = \sum_{i=0}^{k-1} \|\tilde{\Theta}(t+i)\|^2 + K \ln[1 + \|Qx\|^2]$$

Lyapunov function

We have a linear system with

$$\nu[\underline{X}(t)] = -\tilde{\Theta}^T(t) \varphi(t)$$

as a formal input?

ν is bilinear in state vector components?

IDEA # 2

A logarithmic function would dissolve the bilinear product

$$V_x(x(t)) \triangleq \ln[1 + x^T(t) Q^T Q x(t)]$$

Q - weighting matrix

$$V[\underline{X}(t)] = \sum_{i=0}^{k-1} \|\tilde{\Theta}(t+i)\|^2 + \ln[1 + \|Qx\|^2]$$

$$\begin{aligned} V[\underline{X}(t+1)] - V[\underline{X}(t)] &\leq \\ &\leq -\alpha_v \|\tilde{\Theta}(t)\|^2 \cos^2 \alpha(t) \end{aligned}$$

Conclusions - Stability

- Stability in the sense of Lyapunov
- Hard bound on output error from $\|Qx(t)\|^2 < e^{V[\underline{X}(0)]}$
- No assumptions on persistent excitation

CONVERGENCE

• $\hat{\theta}(t), \hat{\theta}(t-1), \dots, \hat{\theta}(t-k+1)$
do not necessarily converge to θ

• $\|x\| \rightarrow 0$

Assume

$$\hat{\theta}(t) = \hat{\theta}(t-k) + \frac{1}{\beta_0 \|\varphi(t-k)\|^2} \varphi(t-k) e_f(t)$$

for all $\|\varphi\| \neq 0$. Then

Theorem 2

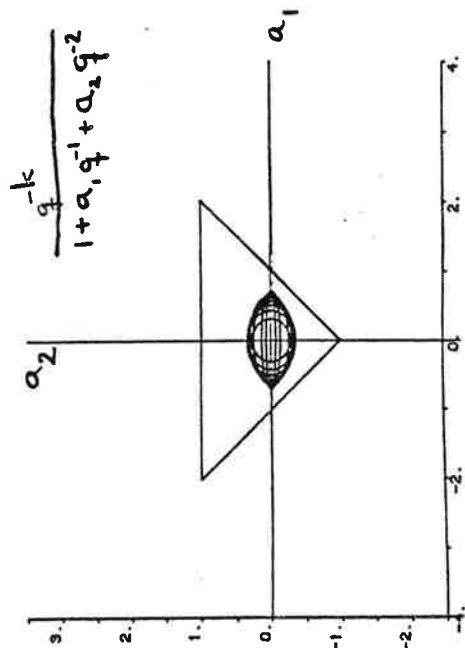
$$\sup \|Qx(t)\|^2 \leq C(t_0) e^{-\delta(t-t_0)}$$

$$V[X(t_0)] < C_0$$

with

$$C(t_0) = \exp[C_0/k]$$

$$\delta \in (0, 1)$$



03.12.08 - 22.12.17 nr. 4
happy Stability region of Lyapunov function

RESTRICTIONS

- Regulators only
- Only well damped closed-loop poles

$$T_1^x A_M^x B_R^x = 1 + p_1 q^{-1} + \dots + p_n q^{-n}$$

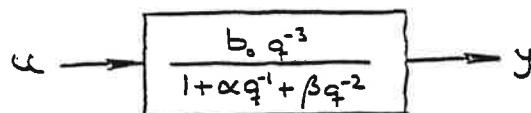
$$\|Q^{-1} p\| < \frac{1}{2} (1 - \rho^2)$$

□ $\hat{\theta}(t) = \hat{\theta}(t-k) + \frac{1}{\beta_0 \|\varphi(t-k)\|^2} \varphi(t-k) e_f(t)$
 \uparrow
 may become large

- Prior knowledge of
 - k - time delay
 - β_0 - estimation of gain

$$0 < \frac{b_0}{\beta_0} < 2$$

Example



Develop an adaptive dead-beat control strategy

$$R^x A^x + S^x B^x = 1$$

$$\begin{cases} A^x(q^{-1}) = 1 + \alpha q^{-1} + \beta q^{-2} \\ B^x(q^{-1}) = b_0 q^{-3} \end{cases}$$

$$\begin{cases} R^x(q^{-1}) = 1 + r_1 q^{-1} + r_2 q^{-2} \\ S^x(q^{-1}) = s_0 + s_1 q^{-1} \end{cases}$$

$$\Rightarrow y = b_0 q^{-3} [u + \theta^T \varphi]$$

$$\begin{cases} \theta = [r_1 \ r_2 \ s_0 \ s_1]^T \\ \varphi(t) = [u(t-1) \ u(t-2) \ y(t) \ y(t-1)] \end{cases}$$

(Ex. :

$$\begin{cases} u(t) = A^x(q^{-1}) \xi(t) = [1 + \alpha q^{-1} + \beta q^{-2}] \xi(t) \\ y(t) = B^x(q^{-1}) \xi(t) = b_0 q^{-3} \xi(t) \end{cases} \quad \text{Ex. 2}$$

→

$$\begin{bmatrix} u(t-1) \\ u(t-2) \\ y(t) \\ y(t-1) \end{bmatrix} = \underbrace{\begin{bmatrix} 1 & \alpha & \beta & 0 \\ 0 & 1 & \alpha & \beta \\ 0 & 0 & b_0 & 0 \\ 0 & 0 & 0 & b_0 \end{bmatrix}}_{M_1} \begin{bmatrix} \xi(t-1) \\ \xi(t-2) \\ \xi(t-3) \\ \xi(t-4) \end{bmatrix}$$

$$\lambda_{\max}(M_1^T M_1) \leq 2(1 + \alpha^2 + \beta^2 + b_0^2)$$

IDENTIFICATION

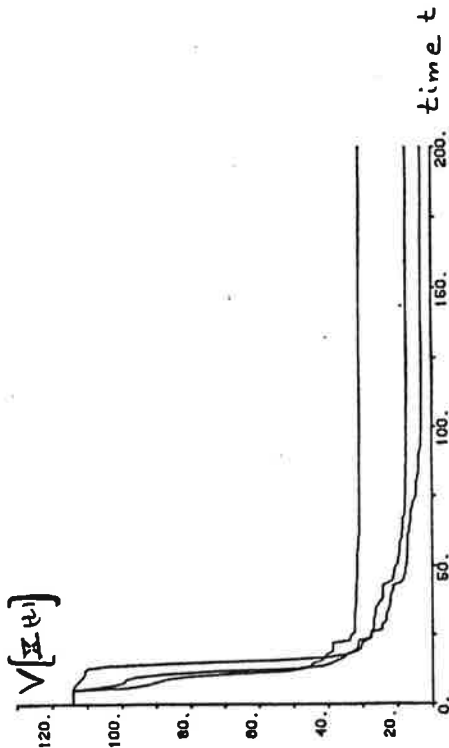
$$\begin{aligned} \hat{\Theta}(t) &= \hat{\Theta}(t-k) + \frac{1}{\beta_3 \|\varphi(t-k)\|^2} \varphi(t-k) e_f(t) \\ e_f(t) &= y(t) - b_0 q^{-k} (u + \hat{\Theta}^T \varphi) \\ &= b_0 q^{-k} (-\tilde{\Theta}^T(t) \varphi(t)) \end{aligned}$$

STATE VECTOR

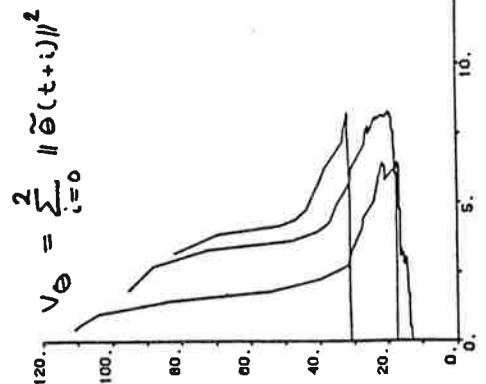
$$[\tilde{\Theta}^T(t+2) \quad \tilde{\Theta}^T(t+1) \quad \tilde{\Theta}^T(t)]^T = z(t)$$

$$V_{\Theta}(z(t)) = z^T(t) z(t) = \sum_{i=0}^2 \|\tilde{\Theta}(t+i)\|^2$$

89.12.09 - 17:54:44 nr: 1
hoopy - Lyapunov Functions x1=0.1, x2=0.5, x3=1.0



89.12.09 - 21:20:33 nr: 1
hoopy - v1h vs. vx



$$V_x = \ln[1 + \|Qx\|^2] \quad \rho = 0.99$$

$$x(t) = [\xi(t-1) \dots \xi(t-7)]^T \quad \text{Ex. 2}$$

$$\begin{aligned} V_x(x(t)) &= \ln[1 + \|Qx(t)\|^2] = \\ &= \ln\left[1 + \sum_{i=1}^7 \xi^2(t-i) \rho^{2(i-1)}\right] \end{aligned}$$

$$X(t) = [\tilde{\Theta}^T(t+2) \dots \tilde{\Theta}^T(t) x^T(t)]^T$$

$$\begin{aligned} V(X(t)) &= \sum_{i=0}^2 \|\tilde{\Theta}(t+i)\|^2 + \\ &+ \ln[1 + \|Qx(t)\|^2] \end{aligned}$$

CONCLUSIONS

- Global stability in the sense of Lyapunov
- Exponential convergence of error (globally)
- What remains?
 - Include reference value
 - LS-identification
 - Include noise and disturbances

Instrumental variable methods for systems operating in closed loop with application to adaptive control

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Instrumental variable methods are normally designed for systems operating in open loop. The reason is that the instruments are typically computed by filtering the input signal. Such an approach does not work if the system operates under feedback control, since then the instruments and disturbances becomes correlated. However, if there is a measurable setpoint (which anyhow is a modest identifiability condition) then this signal can be used for constructions of instruments. Some ways to achieve this are described.

Recent theory of instrumental variable methods show that the estimates are asymptotically Gaussian distributed. The instruments and data prefilters influence the covariance matrix of the estimates. They can in particular be chosen so that an optimal accuracy is obtained. This result is extended to the case of closed loop operation. A key part is then to use the "noise-free" part of the input and the output, i.e. the part that depends of the setpoint but not on the disturbances. The optimal instruments are easily computed once these noise-free signals (or consistent estimates thereof) and the noise correlation properties are known.

An optimal IV method can also be designed on a minimax basis. Minimization is then with respect to the instruments and maximization with respect to the disturbance properties. Also this IV variant is closely related to the noise-free input and output data.

It is further discussed how the above results for a time invariant feedback can be used for design of adaptive control system. Some few numerical simulated examples are also presented.

References

- B. Bauer and H. Unbehauen (1978), On-line Identification of a load- dependent heat exchanger in closed loop using a modified instrumental variable method. Proc. IFAC 7th World Congress, Helsinki.
- T. Söderström and P. Stoica (1983), Instrumental Variable Methods for System Identification. Springer-Verlag, Berlin.
- P. Stoica and T. Söderström (1983), Optimal Instrumental Variable Estimation and Approximate Implementation. IEEE Transactions on Automatic Control, Vol AC-28, pp 757-772.
- E. Trulsson (1983), Adaptive Control Based on Explicit Criterion Minimization. Doctoral Thesis, Department of Electrical Engineering, Linköping University, Linköping, Sweden.

**INSTRUMENTAL VARIABLE METHODS
FOR CLOSED LOOP SYSTEMS**

- MOTIVATION
- IV METHOD
 - DEFINITION
 - CLOSED LOOP SYSTEMS
 - CONSISTENCY
 - EXAMPLES
- OPTIMAL IV METHODS
 - OPTIMAL COV MATRIX
 - MIN MAX OPTIMALITY
 - EXAMPLES
- EXTENSION TO MULTIVARIABLE SYSTEMS
- APPLICATION TO ADAPTIVE CONTROL
 - APPROACHES
 - EXAMPLES
- CONCLUSIONS

MOTIVATIONS

- INSTRUMENTAL VARIABLE METHODS GIVE CONSISTENCY IN PRESENCE OF "ARBITRARY" DISTURBANCES
- POTENTIALLY USEFUL FOR INDIRECT ADAPTIVE CONTROL
 - RECURSIVE IMPLEMENTATION EASY
 - QUICK CONVERGENCE
 - FEW PARAMETERS

BASIC NOTATIONS

SYSTEM (S) $A_0(q^{-1})y(t) = B_0(q^{-1})u(t) + v(t)$
 $v(t) = H(q^{-1})e(t) \quad Ee(t)e(s) = \lambda^2 \delta_{t,s}$

$y(t) = \phi^T(t)\theta_0 + v(t)$

MODEL (M) $A(q^{-1})y(t) = B(q^{-1})u(t)$

$y(t) = \phi^T(t)\theta$

EXPERIMENTAL CONDITION (X)

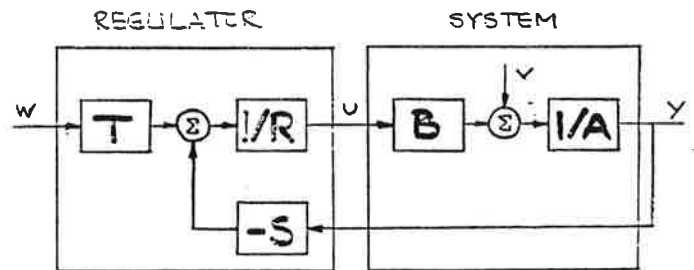
$R(q^{-1})u(t) = -S(q^{-1})y(t) + T(q^{-1})w(t)$

w(t) SETPOINT

A_0, B_0, A, B, R, S, T POLYNOMIALS IN q^{-1}

H RATIONAL FUNCTION IN q^{-1}

$\phi(t) = [-y(t-1) \dots -y(t-na) \quad u(t-1) \dots u(t-nb)]^T$



IV METHOD

$$\hat{\theta} = \left[\frac{1}{N} \sum_{t=1}^N z(t)F(q^{-1})\varphi^T(t) \right]^{-1} \left[\frac{1}{N} \sum_{t=1}^N z(t)F(q^{-1})y(t) \right]$$

- $z(t)$ VECTOR OF INSTRUMENTS
- $F(q^{-1})$ (SCALAR) PREFILTER

EXAMPLES

1) $z(t) = [w(t-1) \dots w(t-na-nb)]^T$

$F(q^{-1}) = 1$

2) $z(t) = [K_1(q^{-1})w(t) \dots K_{na+nb}(q^{-1})w(t)]^T$

CONSISTENCY ANALYSIS

$$\hat{\theta} - \theta_0 = \left[\frac{1}{N} \sum_{t=1}^N z(t)F(q^{-1})\varphi^T(t) \right]^{-1} \left[\frac{1}{N} \sum_{t=1}^N z(t)F(q^{-1})v(t) \right]$$

CONDITIONS

i) $R \triangleq E z(t)F(q^{-1})\varphi^T(t)$
NONSINGULAR

ii) $\theta = E z(t)F(q^{-1})v(t)$

ASSUME

i) $z(t), v(s)$ INDEPENDENT
FOR ALL t AND s

ii) $w(t)$ PERSISTENTLY EXCITING

ACCURACY

$$\sqrt{N}(\hat{\theta} - \theta_0) \xrightarrow{\text{DIST}} N(0, P_{IV})$$

$$P_{IV} = \lambda^2 R^{-1} S R^{-T}$$

$$S = E \{ [F(q^{-1})H(q^{-1})z(t)] [F(q^{-1})H(q^{-1})z(t)]^T \}$$

OPTIMAL COVARIANCE MATRIX

$$P_{IV} \geq P_{IV}^{OPT} = [EH^{-1}(q^{-1})\tilde{\varphi}(t)I][H^{-1}(q^{-1})\tilde{\varphi}(t)]^T$$

$$\tilde{\varphi}(t) = [-\tilde{y}(t-1) \dots -\tilde{y}(t-na) \quad \tilde{u}(t-r) \dots \tilde{u}(t-nb)]^T$$

$\tilde{\varphi}(t)$ DISTURBANCE FREE PART OF $\varphi(t)$

$$\tilde{\varphi}(t) = E[\varphi(t) | w(t-r), w(t-2r) \dots I]$$

$$y(t) = \frac{B(q^{-1})T(q^{-1})}{A(q^{-1})R(q^{-1}) + B(q^{-1})S(q^{-1})} w(t) + \underbrace{\tilde{v}(t)}_{\substack{R(q^{-1}) \\ A(q^{-1})R(q^{-1}) + B(q^{-1})S(q^{-1})}} v(t)$$

EXAMPLE OF OPTIMALITY

SYSTEM $y(t) + ay(t-1) = bu(t-1) + e(t) + ae(t-1)$

REGULATOR (DEADBEAT)

$$u(t) = \frac{a}{a^2} y(t) + \frac{1}{a^2} w(t)$$

$w(t)$ WHITE NOISE $Ew^2(t) = a^2$

⊙ CASE 1

$$z(t) = \begin{bmatrix} w(t-1) \\ w(t-2) \end{bmatrix} \quad P_{IV} = \frac{\lambda Z}{a^2} \begin{bmatrix} 1+a^2 & a^3 b \\ a^2 & (1+a^2)b^2 \end{bmatrix}$$

⊙ CASE 2

OPTIMAL IV $P_{IV} = \frac{\lambda Z}{a^2} \begin{bmatrix} 1-a^2 & 0 \\ 0 & b^2 \end{bmatrix}$

OPTIMAL IV METHODS

⊙ CASE 1: MINIMAL COVARIANCE MATRIX

$$z(t) = H^{-1}(q^{-1})\tilde{\varphi}(t)$$

$$F(q^{-1}) = H^{-1}(q^{-1})$$

⊙ CASE 2: MINIMAX IV METHOD

$$z(t) = \tilde{\varphi}(t) \quad F(q^{-1}) = I$$

SOLUTION TO

MIN	MAX	P_{IV}
$z(t)$	$H(q^{-1})$	
R nonsing	$ H(e^{i\omega}) \leq \alpha$	

EXTENSIONS

⊙ MULTIVARIABLE SYSTEMS

⊙ $nz \geq n\theta$ (OVERDETERMINED IV EQUATIONS)

SYSTEM $y(t) = \theta^T(t)\theta_0 + v(t)$

$$v(t) = H(q^{-1})e(t) \quad Ee(t)e^T(s) = \Lambda \delta_{t,s}$$

$N(\theta - \theta_0) \xrightarrow{DIST} N(0, P_{IV})$

$$P_{IV} \geq P_{IV}^{OPT} = [E[H^{-1}(q^{-1})\tilde{\varphi}^T(t)]^T \Lambda^{-1} [H^{-1}(q^{-1})\tilde{\varphi}^T(t)I]]^{-1}$$

EQUALITY FOR

$$nz = n\theta \quad z(t) = [\Lambda^{-1} H^{-1}(q^{-1})\tilde{\varphi}^T(t)]^T$$

$$F(q^{-1}) = H^{-1}(q^{-1})$$

APPLICATION TO ADAPTIVE CONTROL

ASSUMPTIONS

1. POLE ASSIGNMENT DESIGN
CLOSED LOOP CHAR POL $P_0(q^{-1})$
2. COVARIANCE OPTIMAL IV
WITH $H(q^{-1})$ DESIGN VARIABLE

APPLICATION TO ADAPTIVE CONTROL

ALGORITHM

1. ESTIMATE A AND B BY RECURSIVE IV

$$\hat{A}_t R_t + \hat{B}_t S_t \equiv P_0$$

$$\hat{B}_t(1)T_t = P_0(1)$$

FOR R_t, S_t, T_t

3. COMPUTE $u(t)$ FROM

$$R_t(q^{-1})u(t) = -S_t(q^{-1})y(t) + T_t w(t)$$

4. COMPUTE $\hat{y}(t), \hat{u}(t)$ FROM

$$\hat{y}(t) = \frac{\hat{B}_t(q^{-1})T_t}{P_0(q^{-1})} w(t)$$

$$\hat{u}(t) = \frac{\hat{A}_t(q^{-1})T_t}{P_0(q^{-1})} w(t)$$

(ONLY NUMERATORS ARE TIME VARYING!)

5. COMPUTE $z(t)$ as

$$z(t) = (1-q^{-1})[-\hat{y}(t-1) \dots -\hat{y}(t-na) \hat{u}(t-1) \dots \hat{u}(t-nb)]^T$$

NOTE $H(q^{-1}) = \frac{1}{1-q^{-1}}$

(DRIFT, ACTION FOR RESET WINDUP)

APPLICATION TO ADAPTIVE CONTROL

$$z(t) = H^{-1}[-\hat{y}(t-1) \dots -\hat{y}(t-na) \hat{u}(t-1) \dots \hat{u}(t-nb)]^T$$

$$\hat{y}(t) = \frac{B_0 T}{A_0 R + B_0 S} w(t)$$

$$\hat{u}(t) = \frac{A_0 T}{A_0 R + B_0 S} w(t)$$

GOAL FOR DESIGN

$$A_0 R + B_0 S \equiv P_0$$

$$B_0(1)T(1) = P_0(1)$$

APPLICATION TO ADAPTIVE CONTROL EXAMPLES

CASE 1

$$S \quad y(t) - 1.9y(t-1) + 0.9y(t-2) = 1.0u(t-1) + 0.5u(t-2)$$

$$M \quad y(t) = a_1 y(t-1) + a_2 y(t-2) = b_1 u(t-1) + b_2 u(t-2)$$

CLOSED LOOP POLES: 0.5, 0, 0

CASE 2

$$S \quad y(t) - 0.8y(t-1) = 1.0u(t-1) + \frac{1}{1-q^{-1}} e(t)$$

$$\lambda_e = 0.05$$

$$M \quad y(t) - 0.8y(t-1) = bu(t-1)$$

CLOSED LOOP POLE: 0

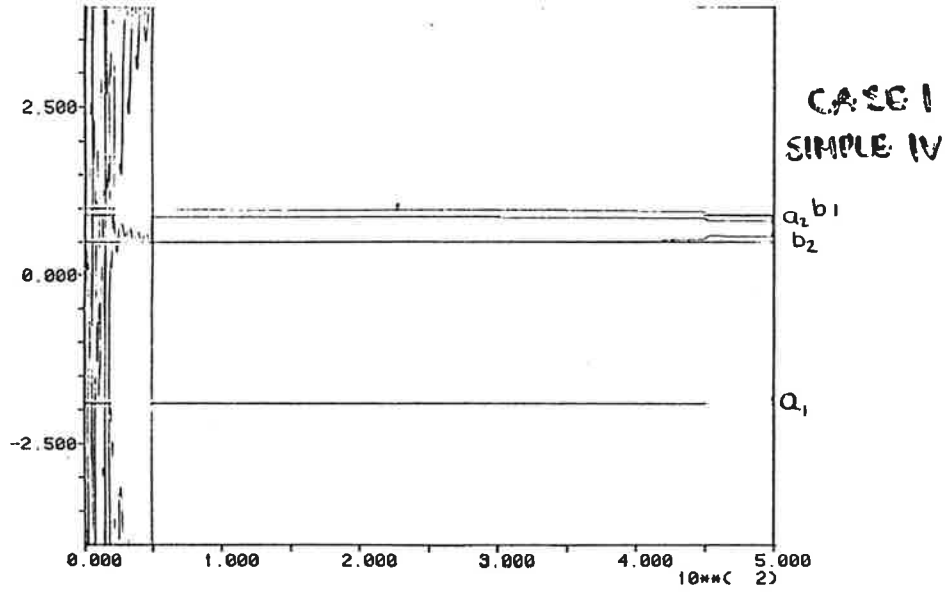
METHOD

$$\bullet I_1 \quad z(t) = (w(t-1) \dots w(t-na-na))I^T \quad F(q^{-1}) = 1 \quad \text{SIMPLE IV}$$

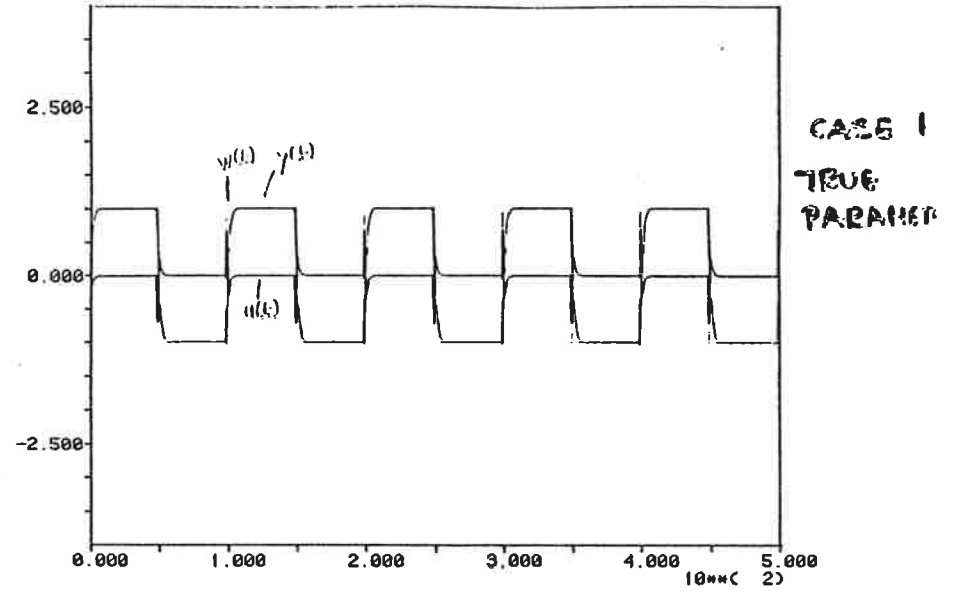
$$\bullet I_2 \quad z(t) = (1-q^{-1})\hat{\phi}(t) \quad F(q^{-1}) = 1-q^{-1} \quad \text{COVARIANCE OPTIMAL IV}$$

$$\bullet I_3 \quad z(t) = \hat{\phi}(t) \quad F(q^{-1}) = 1 \quad \text{MINIMAX IV}$$

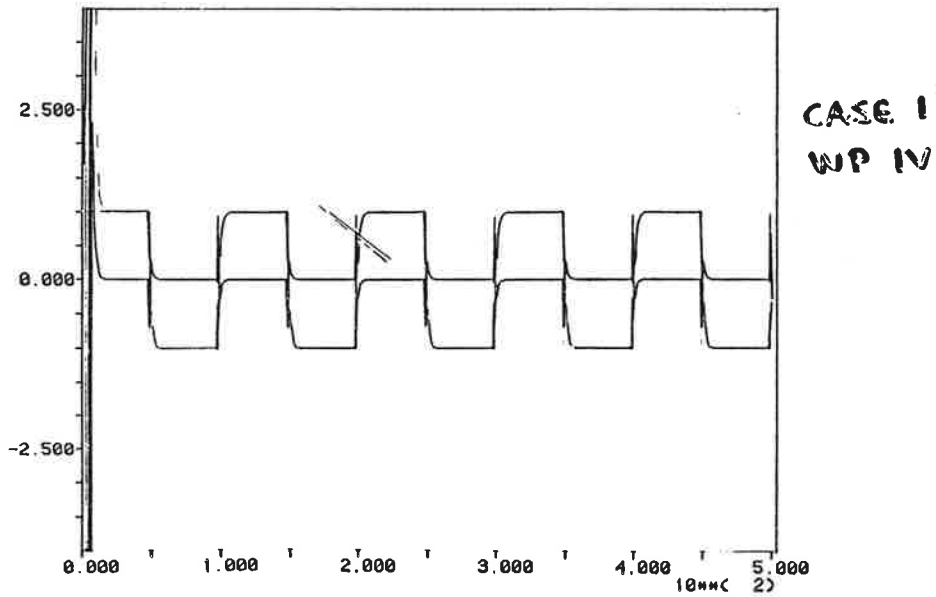
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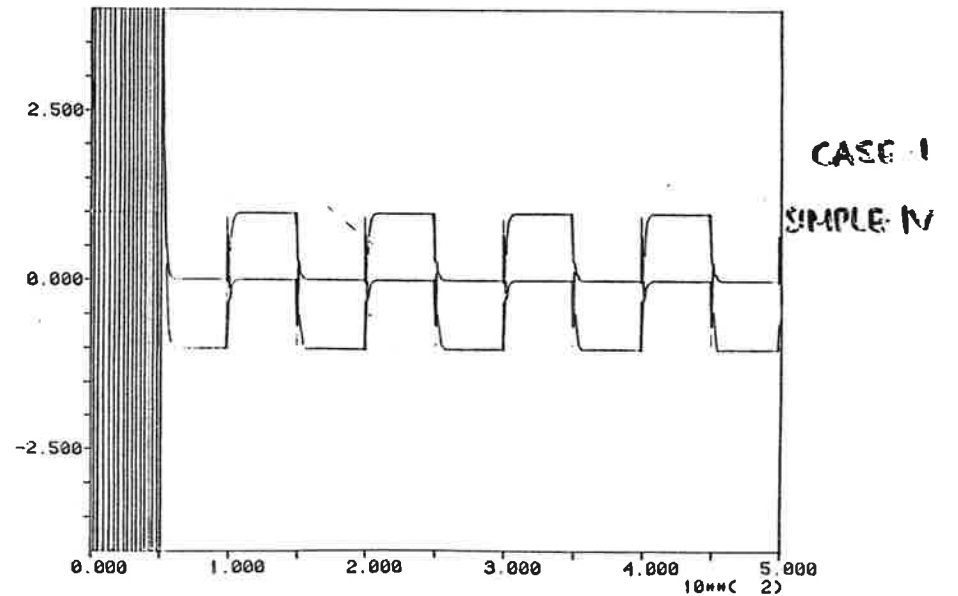
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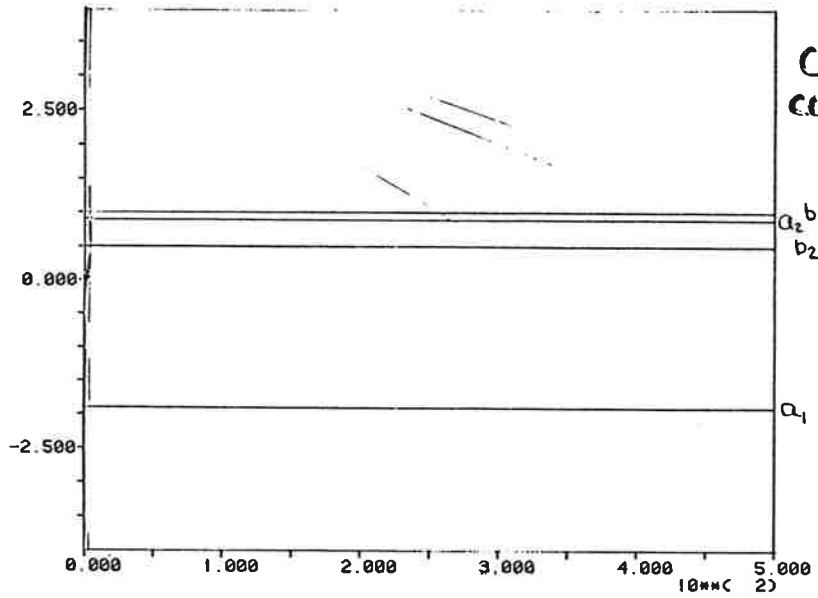
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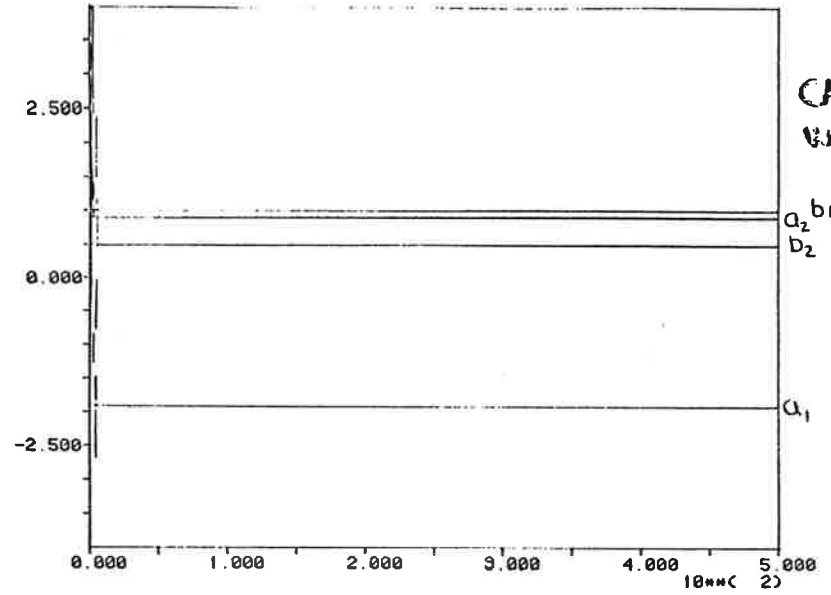


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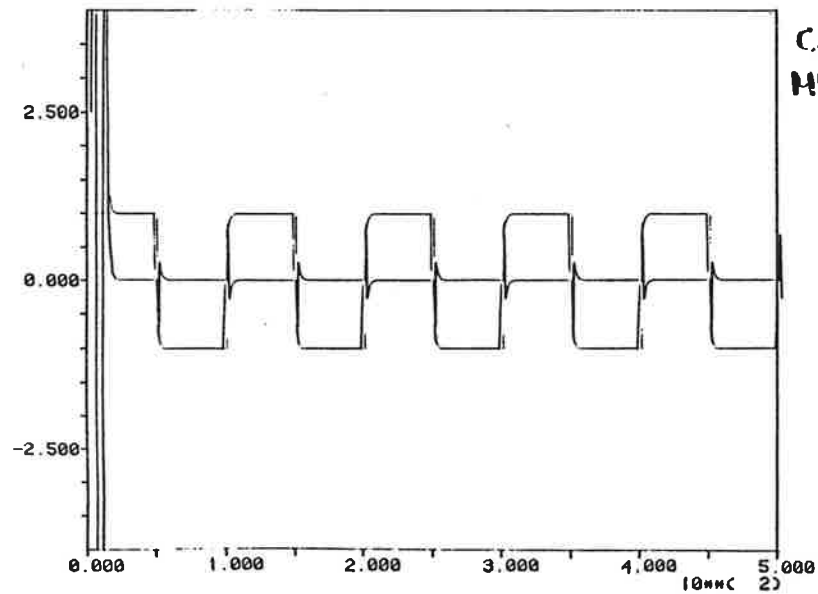
CASE 1
CONV OPT II.

IEX=2 IEST=2 AL= 0.00 ALAMBDA=1.00 P0=10.00 ULIMIT=10.00



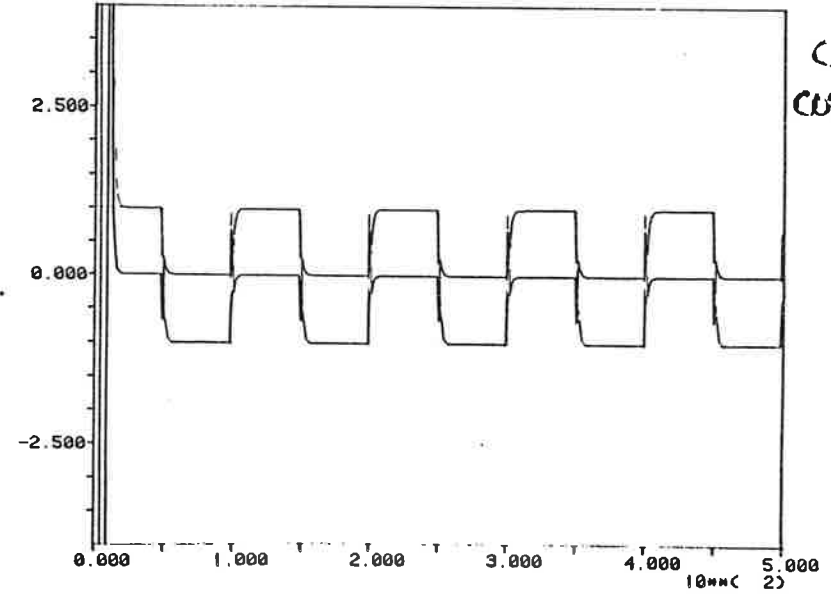
CASE 1
WP IV

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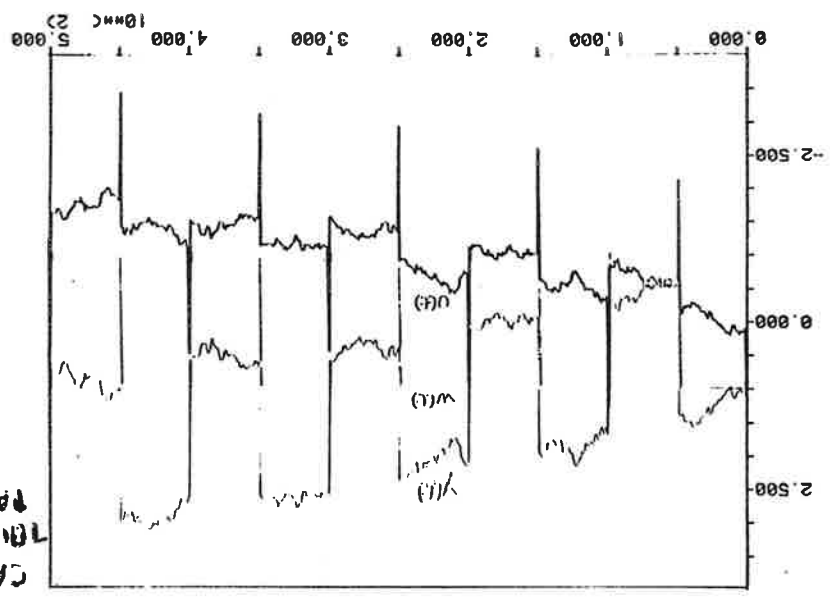
CASE 1
MINMAX II.

IEX=2 IEST=3 AL= 0.00 ALAMBDA=1.00 P0=10.00 ULIMIT=10.00



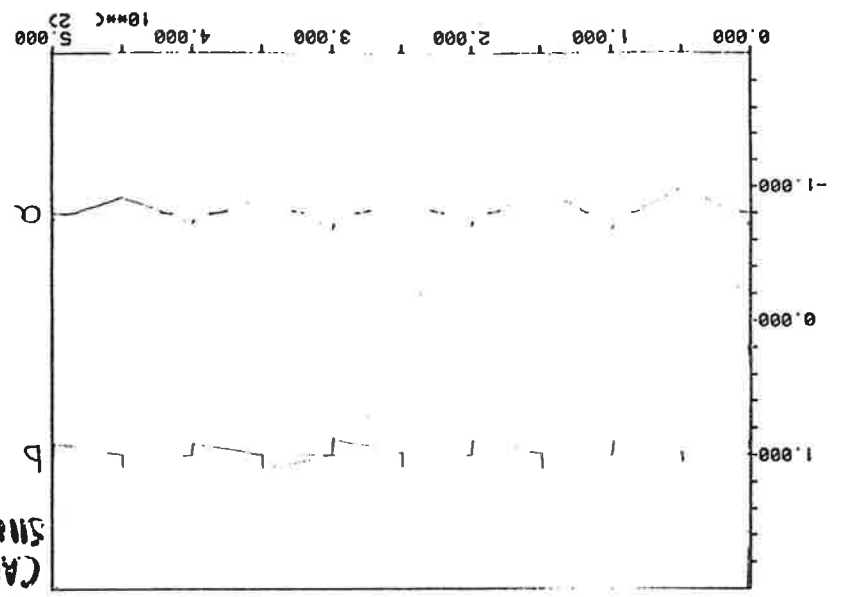
CASE 1
CONV OPT I.

CASE 2
 TONG
 PARAMETER



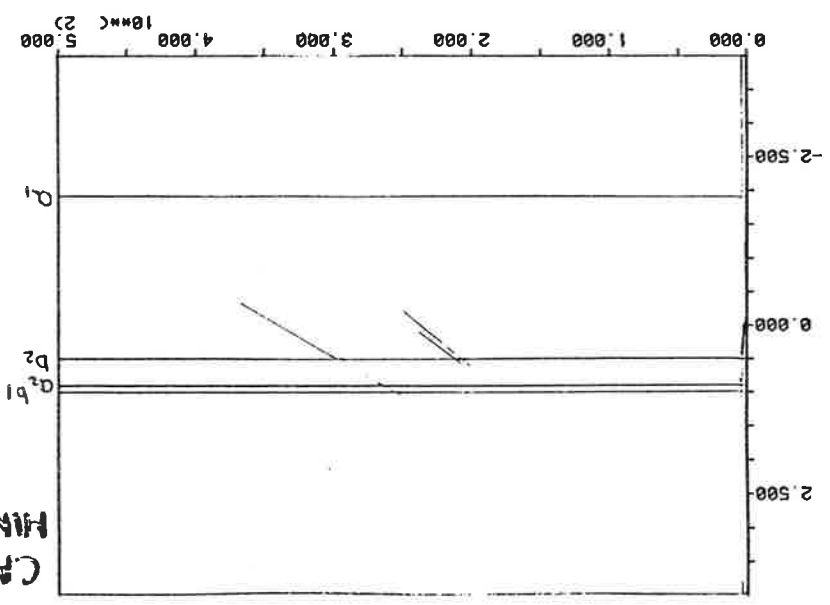
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CASE 2
 SINGLE IV



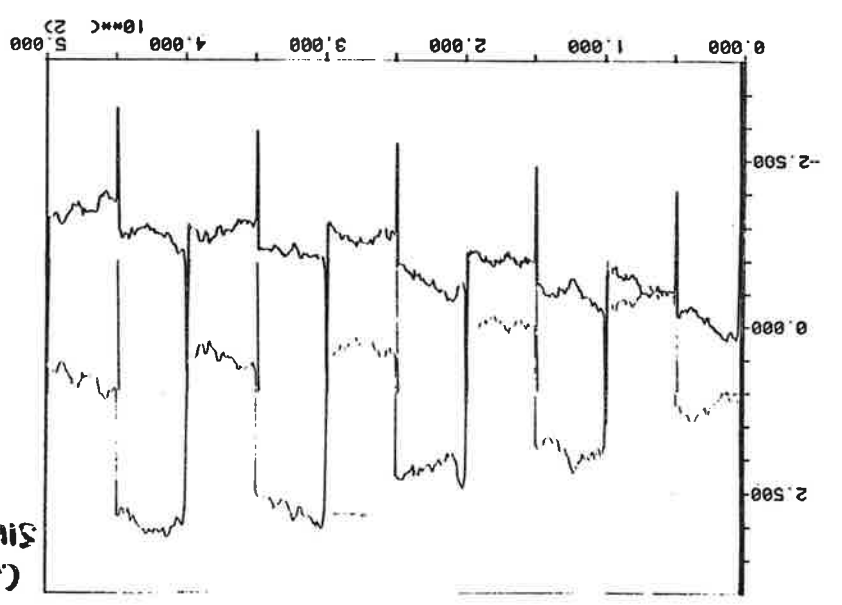
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CASE 1
 MINIMAX

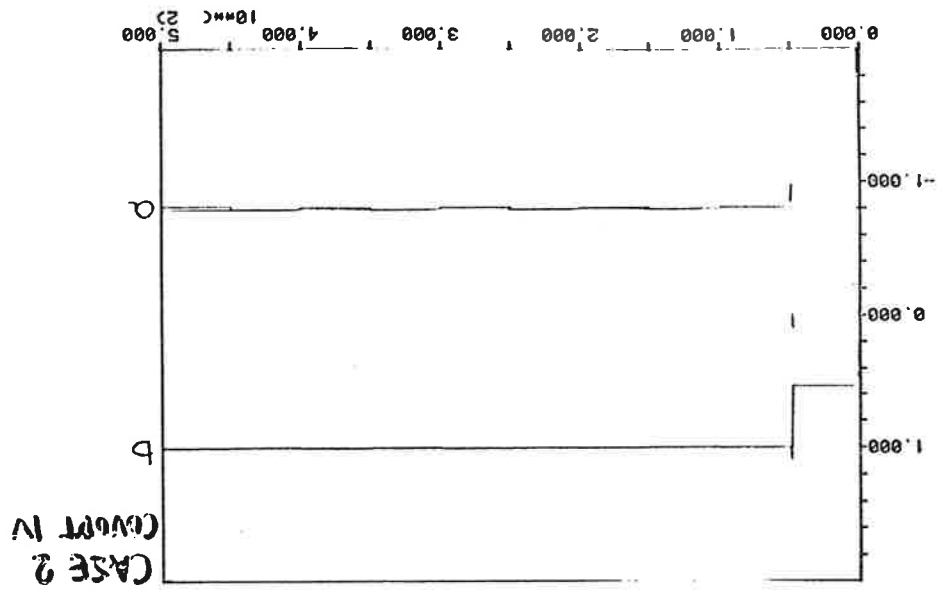


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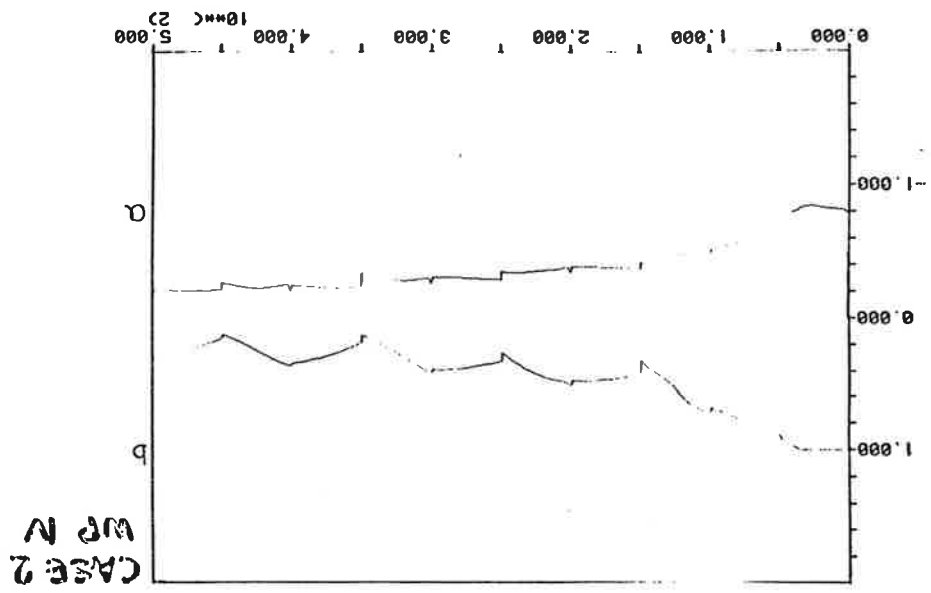
CASE 2
 SINGLE IV



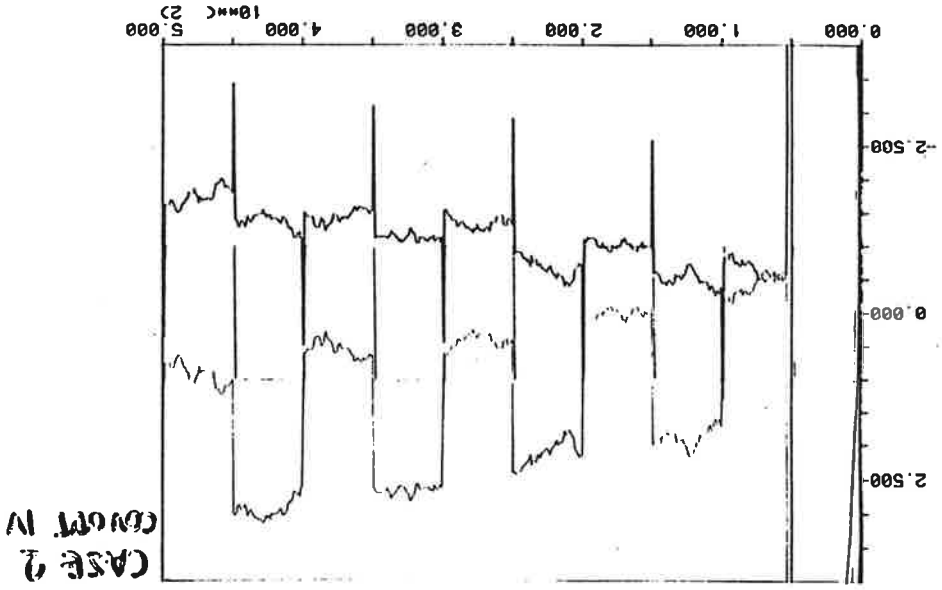
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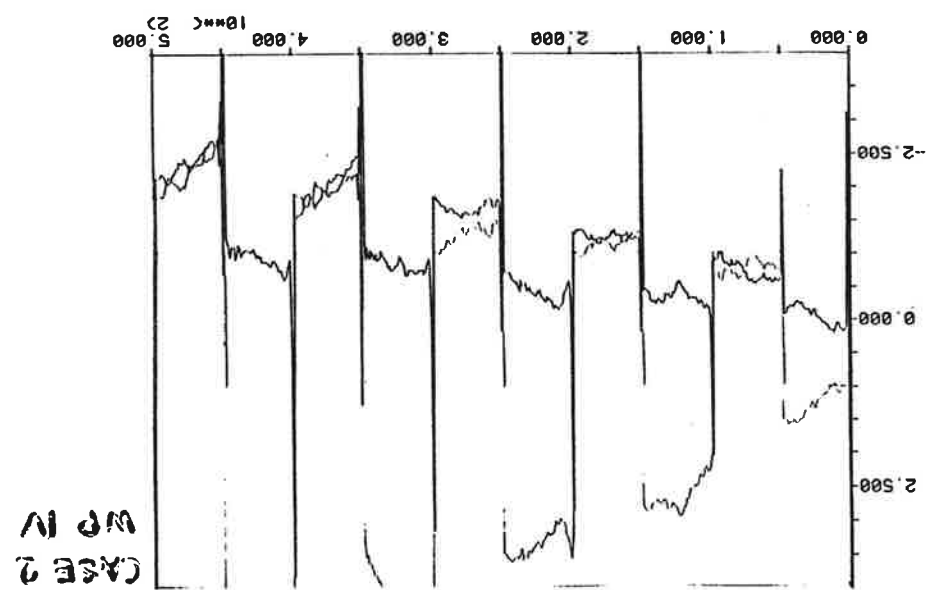
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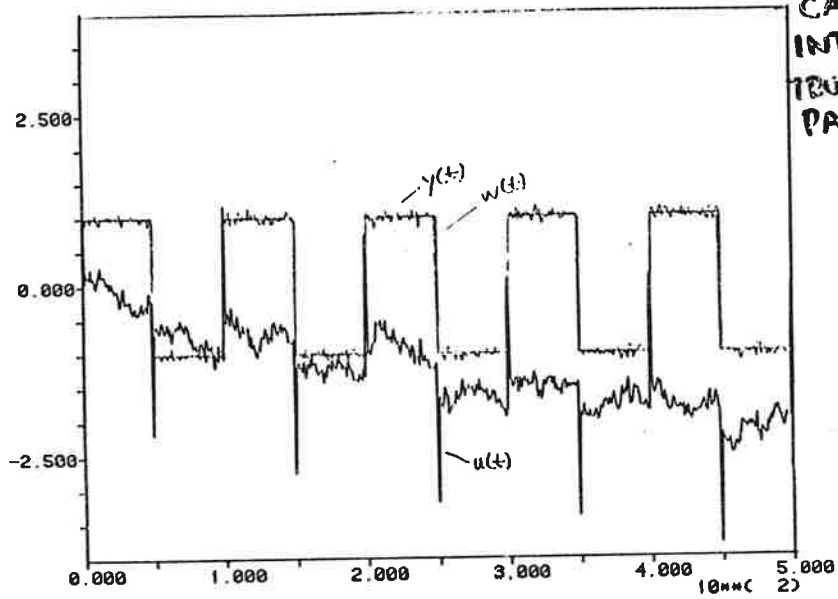


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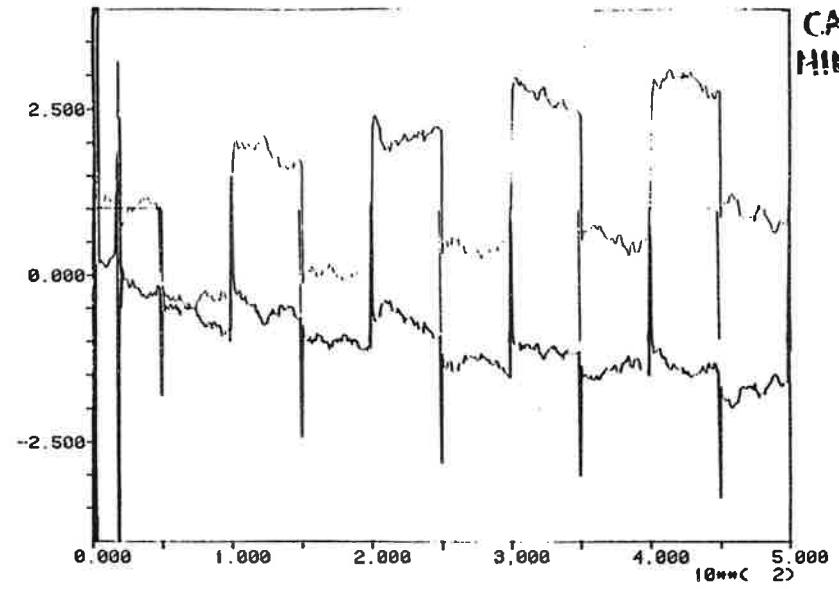
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IEX=3 IEST=4 AL= 0.05 ALAMBDA=1.00 F0= 0.00 ULIMIT=10.00



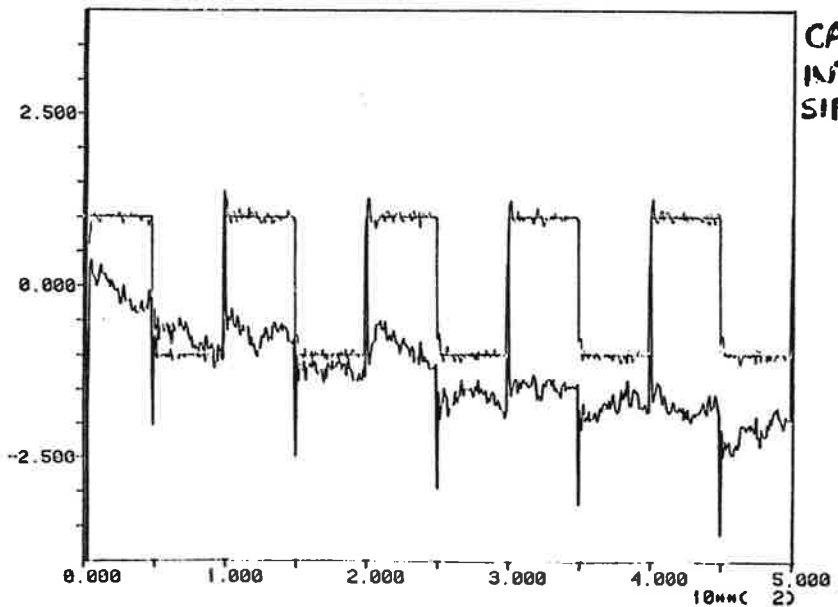
CASE 1
INT. REG.
TRUE
PARAMETER

IEX=3 IEST=4 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



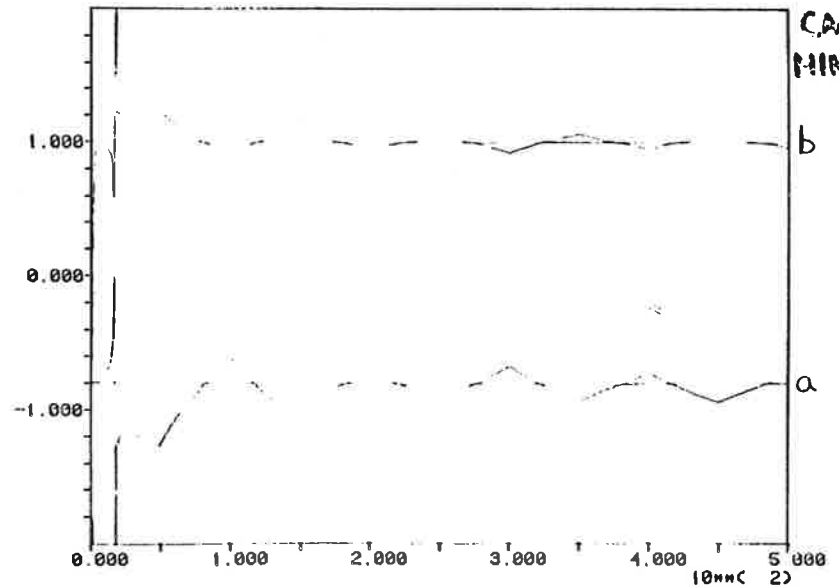
CASE 2
MINMAX 1

IEX=3 IEST=1 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



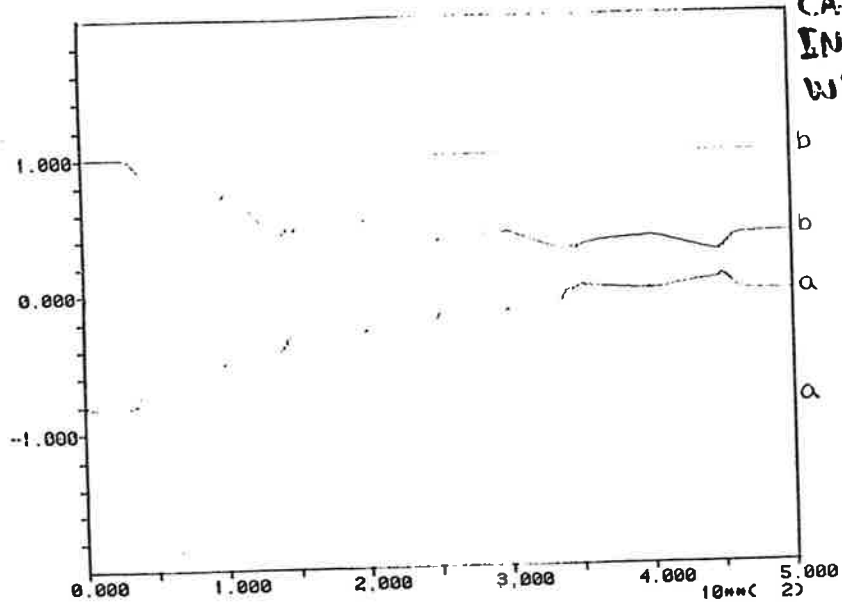
CASE 2
INT. REG.
SIMPLE IV

IEX=3 IEST=4 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



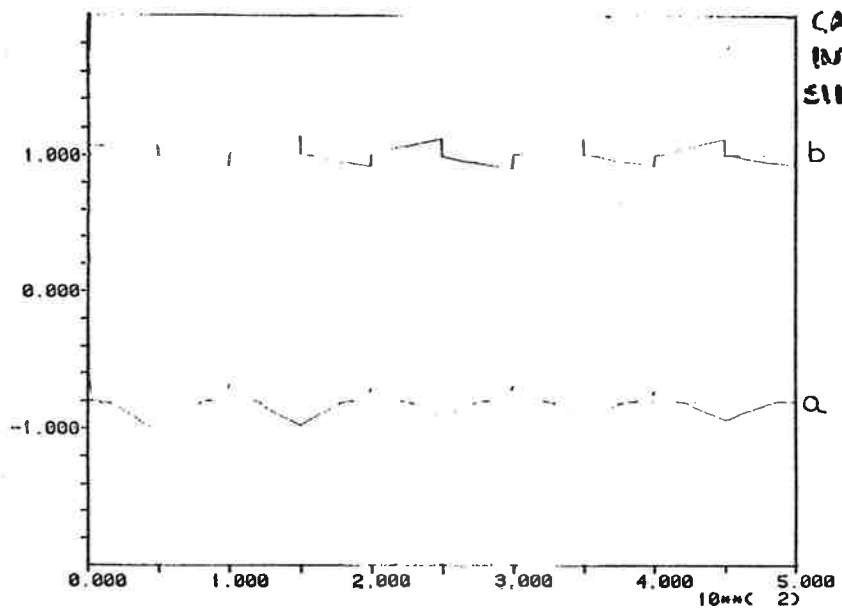
CASE 2
MINMAX 1

IEX=3 IEST=2 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



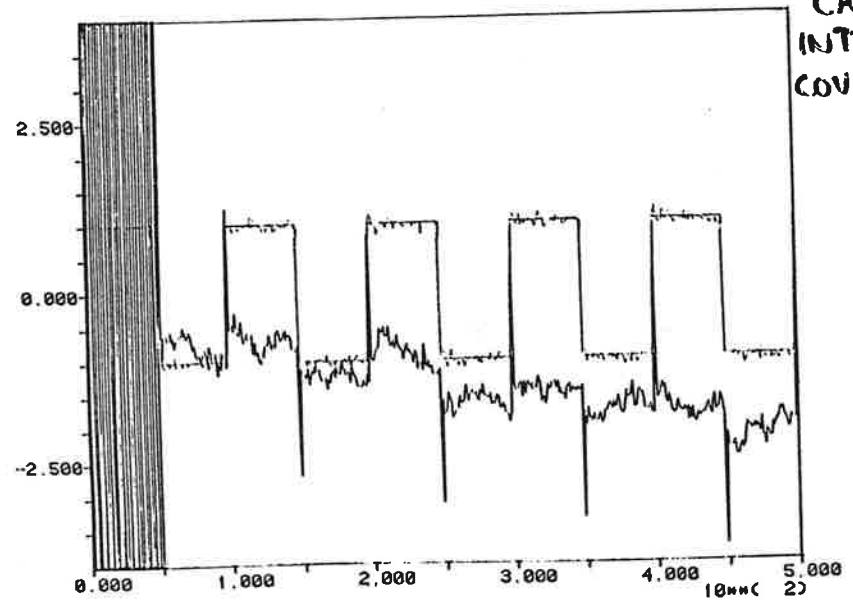
CASE 2
INT. REG
WP IV

IEX=3 IEST=1 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



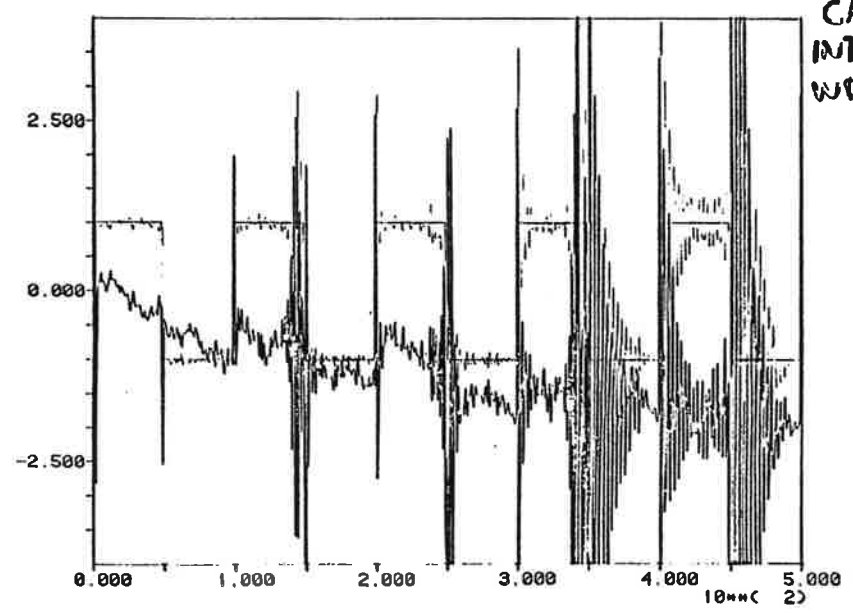
CASE 2
INT. REG
SIMPLE II

IEX=3 IEST=3 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00



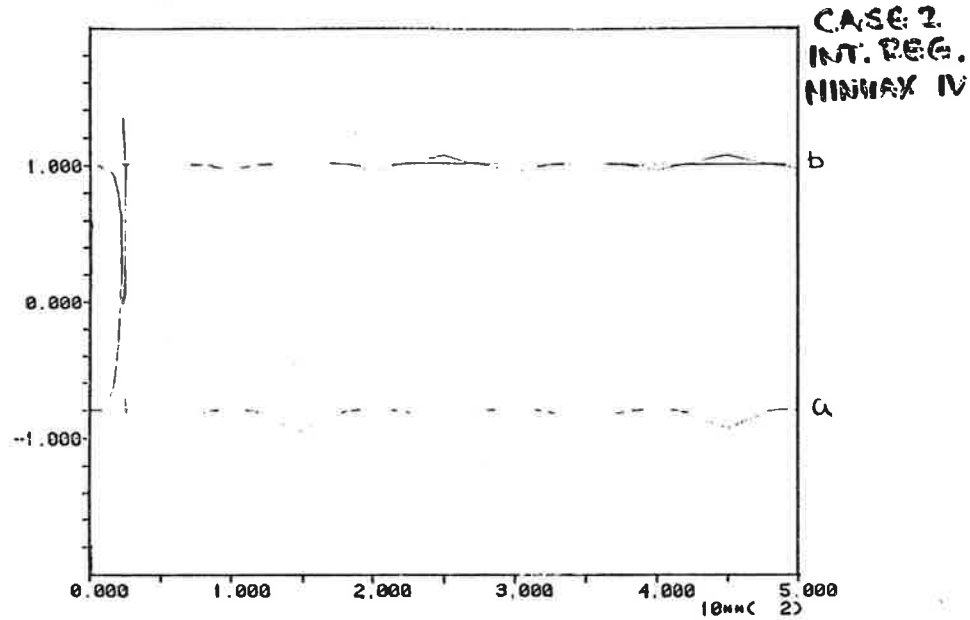
CASE 2
INT. REG
CONV OPT IV

IEX=3 IEST=2 AL= 0.05 ALAMBDA=1.00 F0=10.00 ULIMIT=10.00

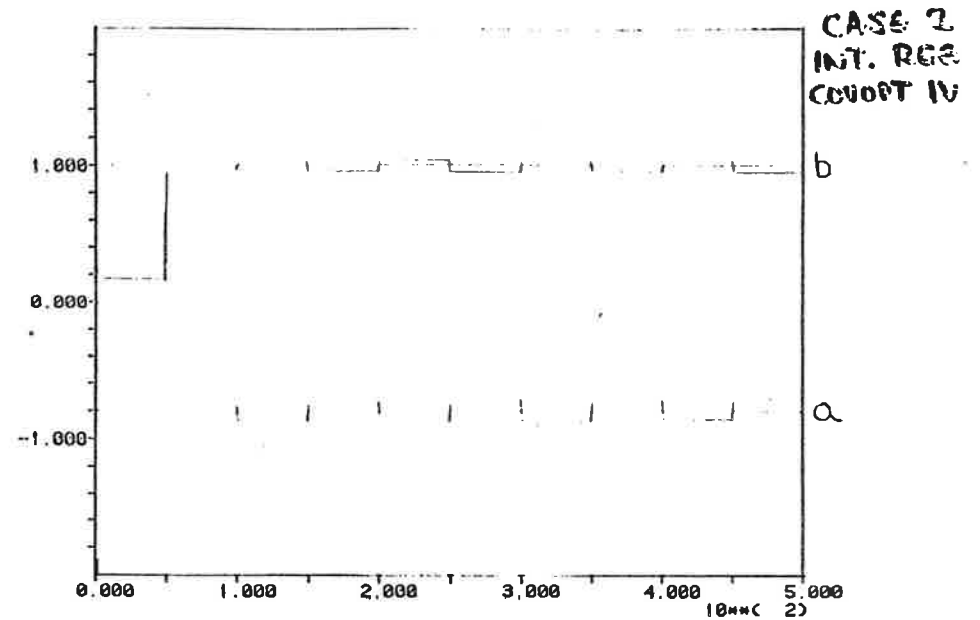


CASE 2
INT. REG
WP IV

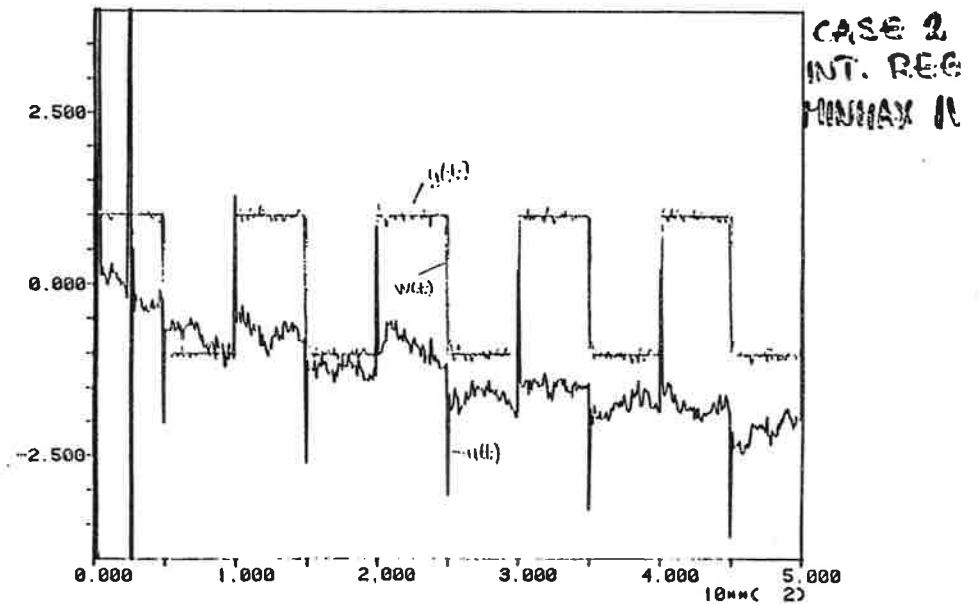
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IEX=3 IEST=3 AL= 0.05 ALAMBDA=1.00 P0=10.00 ULIMIT=10.00



IEX=3 IEST=4 AL= 0.05 ALAMBDA=1.00 P0=10.00 ULIMIT=10.00



CONCLUSIONS

- ① IV METHODS POSSIBLE TO USE IN CLOSED LOOP
- ② CHOICE OF INSTRUMENTS CAN BE IMPORTANT
- ③ OPTIMAL IV METHODS EXIST
- ④ SOME POTENTIAL FOR DESIGN OF ADAPTIVE CONTROL

Adaptive spectral factorization

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Despite the great amount of work in the last ten years on the behaviour of various recursive parameter estimation and self-tuning control schemes the situation is disappointing. though a number of global convergence results are available they usually entail restrictions that amount to knowledge of the true system.

Thus, the positive real condition cannot be checked unless the true system parameters are known. Further, monitoring schemes cannot be properly designed unless the true system parameters are known. Also, even for the globally convergent algorithms, various internal filters are not guaranteed to be stable.

In this work some building blocks for algorithm design are suggested. The Levinson, Burg or Lattice algorithm guarantees stability for autoregressive (AR) models. Wilson's (1969) Newton Raphson scheme for spectral factorization guarantees stability of the spectral factor iterate at each iteration. Finally, general regression enjoys a bounded posterior error power property. The use of these building blocks together with the idea of split recursions is illustrated by developing a number of algorithms for ARMA and ARMAX recursive estimation (no iteration is involved).

One of those schemes (called RF_2) is shown to be globally convergent. This scheme is then used to develop a convergent self-tuning Kalman filter. The algorithm is free of the criticism mentioned above.

Finally, the three tools above are combined to produce a self-tuning LQG controller. It enjoys some stability properties but no convergence proof is available.

Reference

G.T. Wilson (1969). Factorization of the covariance generating function of a pure moving average process, SIAM J. Numer. Anal., 6, 1-7.

ADAPTIVE SPECTRAL FACTORIZATION

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2

MODELS

$$(L \equiv z^{-1} \equiv q^{-1})$$

1) ARMA

$$A(z)Y_k = D(z)E_k$$

$$A(z) = 1 + a_1 z^{-1} + \dots + a_p z^{-p}$$

$$\Leftrightarrow Y_k = \phi_k' \theta + \epsilon_k$$

$$\theta' = (a', d')$$

$$\phi_k' = (-y_{k-1}, \dots, y_{k-p})$$

2) ARMAX OR CARMA

$$A(z)Y_k = b(z)U_k + c(z)E_k$$

3) ARX OR CAR (BUT NOT ARMA)

$$A(z)Y_k = b(z)U_k + \epsilon_k$$

4) TFARMA

$$Y_k = \frac{b(z)}{A(z)} U_k + \frac{D(z)}{C(z)} E_k$$

5) FIR OR REGRESSION

$$Y_k = b(z)U_k + \epsilon_k$$

• REVIEW

- 1 MODELS
- 2 RECURSIVE PARAMETER ESTIMATION
- 3 STABILITY OF ALGORITHMS
- 4 SPECTRAL FACTORIZATION

• PRESENT

- 1 FOUR RECURSIVE SCHEMES (SPLIT RECURSIONS)
- 2 ONE CONVERGES GLOBALLY
- 2 A CONVERGENT SELF TUNING KALMAN FILTER
- 3 USE IN SELF TUNING CONTROL (LQG)
- 1 A GENERAL MONITORING SCHEME

3

RECURSIVE PARAMETER ESTIMATION

$$\theta_{k+1} = F_k \theta_k + P_k \psi_k \epsilon_k$$

LONG MEMORY i.e. GAIN = $P_k \rightarrow 0$

OR RECURSIVE OR SELF-TUNING

SHORT MEMORY i.e. GAIN = $P_k > P > 0$

• WAYS TO GENERATE GAIN & GRADIENT

(1) PER

(2) PLR

(3) EKF

(4) IV

NOTE

SPLIT RECURSIONS

of YOUNG'S

IV-AML

• AIMS FOR AN ALGORITHM

STABILITY e.g. ERROR POWER BOUNDED

CONVERGENCE e.g. ERROR POWER \rightarrow INNOVATIONS

EFFICIENCY e.g. CRAMER-RAO BOUND

STABILITY PROPERTIES OF SOME SIGNAL PROCESSING ALGORITHMS

(1) LEVINSON

OR BURG OR LATTICE

ENSURE AR POLYNOMIAL IS STABLE

(2) WILSON'S ALGORITHM FOR SPECTRAL FACTORISATION

SOLVE $|X(z)|^2 = \sum_{k=0}^p x_k z^k$

AT EACH ITERATION $D_k(z)$ IS STABLE

UNTRUE? OF BAUER

(3) REGRESSION ERROR POWER \leq DATA POWER

DATA y_k REGRESSORS x_k

$$b_k = a_{k-1} + P_k x_k c_k \quad c_k = y_k - x_k' a_{k-1}$$

$$P_k^{-1} = P_{k-1}^{-1} + x_k x_k' \quad \eta_k = y_k - x_k' a_k$$

$$\Rightarrow \lim_{n \rightarrow \infty} \frac{1}{n} \sum_{k=1}^n \eta_k^2 \leq \lim_{n \rightarrow \infty} \frac{1}{n} \sum_{k=1}^n y_k^2$$

PROOF OF STABILITY PROPERTY OF REGRESSION

$$a_k = a_{k-1} + P_k x_k c_k \quad c_k = y_k - x_k' a_{k-1}$$

$$\Rightarrow P_k^{-1} a_k = P_{k-1}^{-1} a_{k-1} + x_k y_k = P_{k-1}^{-1} a_{k-1} + x_k c_k$$

$$\Rightarrow T_k = a_k' P_k^{-1} a_k = T_{k-1} - a_{k-1}' x_k x_k' + y_k^2 + x_k' P_{k-1} x_k c_k^2 = T_{k-1} + (y_k - c_k)^2 + x_k' P_{k-1} x_k c_k^2 = T_{k-1} + y_k^2 - c_k^2 (1 - x_k' P_{k-1} x_k)$$

SUM \Rightarrow

$$T_n + \sum_{k=1}^n c_k^2 (1 - x_k' P_{k-1} x_k) = \sum_{k=1}^n y_k^2$$

BUT

$$\eta_k = y_k - a_k' x_k = y_k - a_{k-1}' x_k - x_k' P_k x_k c_k = y_k - a_{k-1}' x_k - x_k' P_k x_k c_k = y_k - a_{k-1}' x_k - x_k' P_k x_k c_k$$

$$\Rightarrow T_n + \sum_{k=1}^n \frac{\eta_k^2}{1 - x_k' P_k x_k} = \sum_{k=1}^n y_k^2$$

BUT $x_k' P_k x_k \leq 1$

SPECTRAL FACTORISATION

$$|z|^2 = \frac{\sigma^2}{y_0} \left| 1 + \sum_{k=1}^p d_k z^k \right|^2 = 1 + \sum_{k=1}^p b_k z^k + \sum_{k=1}^p f_k z^{-k}$$

EQUIVALENTLY

$$\sqrt{|1 + h'(zI - F)g|} = 1 + m'(zI - F)^{-1}h \cdot m'(z^{-1}I - F')^{-1}h$$

$$F = \begin{bmatrix} 0 & 1 & 0 & \dots & 0 \\ 0 & 0 & 1 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \dots & 1 \\ 0 & 0 & 0 & \dots & 0 \end{bmatrix} \quad h = \begin{bmatrix} 1 \\ 0 \\ \vdots \\ 0 \end{bmatrix} \quad m = \begin{bmatrix} p_1 \\ \vdots \\ p_p \end{bmatrix} \quad g = \begin{bmatrix} d_1 \\ \vdots \\ d_p \end{bmatrix}$$

STATE SPACE VERSION IS POSITIVE REAL EQUATIONS

$$P = F P F' + g g g'$$

$$z = m - F P h$$

$$V = 1 - h' P h$$

ALGORITHMIC SOLUTIONS

(a) LINEARLY CONVERGENT (BAUER, ...)

(b) QUADRATICALLY CONVERGENT (WILSON, HENNER)

OFFLINE SF - LINEAR SCHEME

FREQUENCY DOMAIN - BAUER

$$\delta_0^{(0)} = P_0$$

$$\delta_0^{(j)} = P_0 - \sum_{k=1}^j \frac{\delta_0^{(k-1)} \delta_0^{(j-k)}}{\delta_0^{(j-1)}} \quad 1 \leq j \leq p$$

$$\delta_0^{(p)} = 1 - \sum_{k=1}^p (\delta_0^{(k)})^2 / \delta_0^{(p-k)}$$

TIME DOMAIN (STATE SPACE)

$$\hat{P}_{k+1} = F \hat{P}_k F' + g_k V_k^{-1} g_k \quad ; \quad \hat{P}_0 = 0$$

$$g_k = m - F \hat{P}_k h$$

$$V_k = 1 - h' \hat{P}_k h$$

$$\delta_k(z) = V_k (1 + h'(zI - F)^{-1} g_k V_k^{-1})$$

OFFLINE SF - QUADRATIC SCHEMES

• FREQUENCY DOMAIN (WILSON, 1969)

SOLVE $\frac{\hat{\delta}_j(z)}{\delta_j(z)} + \frac{\hat{\delta}_j(z^{-1})}{\delta_j(z^{-1})} = \frac{2 \sum \beta_i z^i}{\delta_j(z) \delta_j(z^{-1})}$

$\hat{\delta}_{j,h}(z) = \frac{1}{2} \delta_j(z) + \frac{1}{2} \hat{\delta}_{j,h}(z)$

$\Leftrightarrow T(\underline{s}_j) \hat{\underline{s}}_j = 2 \begin{pmatrix} 1 \\ \underline{m} \end{pmatrix}$

$\hat{\underline{s}}_{j,h} = \frac{1}{2} \hat{\underline{s}}_j + \frac{1}{2} \hat{\underline{s}}_j$

$T(\underline{s}) = \begin{bmatrix} \delta_0 & \delta_1 & \delta_p \\ \delta_1 & \delta_0 & 0 \\ \delta_p & 0 & 0 \end{bmatrix} + \begin{bmatrix} \delta_0 & \delta_1 & \delta_p \\ 0 & 0 & \delta_0 \end{bmatrix}$

• TIME DOMAIN (STATE SPACE) HEWER, 1973

$\hat{P}_{k+1} = F \hat{P}_k F' + g_k V_k^{-1} g_k' \quad ; \hat{P}_0 = 0$

$g_k = M - F \hat{P}_k h$

$V_k = 1 - h' \hat{P}_k h$

$\delta_k(z) = V_k (1 - h' z I - F)^{-1} g_k V_k^{-1}$

10

RHF₁

$A(z^{-1}) y_k = w_k = D(z^{-1}) e_k$

(a) IV (= Hankel) ESTIMATION OF \underline{a}_k

(b) GENERATE $\hat{w}_k = \hat{A}_k(z^{-1}) y_k$

UPDATE $\hat{\delta}_{k+1}^{(k+1)}(z) = \hat{\delta}_{k+1}^{(k)}(z) - \frac{1}{k+1} (\hat{w}_{k+1} - \hat{\delta}_{k+1}^{(k)}(z))$

DO ONE STEP OF OAVCA

$\hat{\delta}_p^{(k+1)} = \hat{\delta}_p^{(k+1)}$

$\hat{\delta}_1^{(k+1)} = \hat{\delta}_1^{(k+1)}(z) = \sum_{l=1}^p \hat{\delta}_l^{(k+1)} \hat{\delta}_{l-1}^{(k+1)} / \hat{\delta}_0^{(k+1)}$

$\hat{\delta}_0^{(k+1)} = 1 - \sum_{l=1}^p (\hat{\delta}_l^{(k+1)})^2 / \hat{\delta}_0^{(k+1)}$

OR STATE SPACE

$\hat{P}_{k+1} = F \hat{P}_k F' + g_k V_k^{-1} g_k'$

$g_k = \begin{pmatrix} m_k \end{pmatrix} - F \hat{P}_k h$

$m_k = [P_0^{(k)} \dots P_p^{(k)}]'$

$V_k = 1 - h' \hat{P}_k h$

DERIVATION + PROPERTIES OF WILSON'S ALGORITHM

• GIVEN D_p CHOOSE $D_2 = D_1 + dD_1 \Leftrightarrow$

$|D_2|^2 = |D_1 + dD_1|^2 \approx P = \sum \beta_i z^i$

SO SET $d_0, d_1 + d d_0, d_1 + d d_0, -P$

$\Rightarrow d_0/d_1 + d d_1/d_1 = P/|D_1|^2 - 1$

• STABILITY

BUT $|D_2|^2 = P + |dD_1|^2 \gg |dD_1|^2$

$= |D_2 - D_1|^2 \quad \text{ON } |z|=1$

ROUGHLY $\Rightarrow D_2, D_1$ HAVE SAME

NO. OF ZEROS INSIDE $|z|=1$

• NOTE SET

$\hat{d}_1/d_1 + \hat{d}_1/d_1 = 2P/|D_1|^2$

$\Rightarrow D_2 = \frac{1}{2} D_1 + \frac{1}{2} \hat{D}_1$

11

RHF₂

$A(z^{-1}) y_k = w_k = D(z^{-1}) e_k$

(a) IV (= Hankel) ESTIMATION OF \underline{a}

(b) GENERATE $\hat{w}_k = \hat{A}_k(z^{-1}) y_k$

UPDATE $\hat{\delta}_k^{(k+1)}(z)$

DO ONE STEP OF WILSON

$\hat{\underline{s}}_k = T(\underline{s}_k) \begin{bmatrix} 1 \\ \underline{m}_k \end{bmatrix}$

$\hat{\underline{s}}_{k+1} = \frac{1}{2} \hat{\underline{s}}_k + \frac{1}{2} \hat{\underline{s}}_k$

OR HEWER

$\hat{P}_{k+1} = F \hat{P}_k F' + g_k V_k^{-1} g_k'$

$g_k = \begin{pmatrix} m_k \end{pmatrix} - F \hat{P}_k h$

$V_k = 1 - h' \hat{P}_k h$

TROUBLE WITH ON-LINE

SF₂

RECALL $D_2 = |D_2 - D_1|^2 + P$

NEED $P > 0$ TO USE ROUCHE

NO GUARANTEE e.g.

$1 + \alpha P_1 \cos \omega \geq 0 \quad \forall \omega$ iff $|P_1| \leq \frac{1}{2}$

SO USE A MONITORING SCHEME

1) GET $\hat{\delta}_{j+1}$ AND $\hat{\delta}_{j+1} = \frac{1}{2} \hat{\delta}_j + \frac{1}{2} \hat{\delta}'_j$

2) STABILITY CHECK OF $\hat{\delta}_{j+1}$

3) IF IT FAILS FORM

$$\hat{\beta} = |\hat{\delta}_{j+1}|^2$$

AND REO ONE STEP OF SF₂

THEN $\hat{\delta}_{j+1} = \frac{1}{2} \hat{\delta}_j + \frac{1}{2} \hat{\delta}'_j$

IS GUARANTEED STABLE

• DOES NOT AFFECT CONVERGENCE

PROPERTY / AR	AR STABLE	MA STABLE	TRANSIENTS	CONVERGES
-LG.				
RHF ₁				
RHF ₂		✓		✓
RLF ₁	✓			
RLF ₂	✓	✓	✓	

- PARASITICS — RLF₂ ✓
- TIME VARYING PARAMETERS
ALL EASILY MODIFIED
ONLY RLF₂ RETAINS PROPERTIES
- GENERAL MONITORING METHOD

RLF

(a) GIVEN $a_k(z^{-1})$

FORM $\hat{w}_k = (1 + \alpha_k(z^{-1})) y_k$ AND $\hat{w}_k^{(1)} = 1 = 0 \dots P$

F₁ DO ONE STEP OF BAUER

F₂ DO ONE STEP OF WILSON

GIVES $d_{k+1}(z^{-1})$

(b) WITH $d_{k+1}(z^{-1})$

FORM $\hat{e}_{k+1} = y_{k+1} - d_{k+1}(z^{-1}) \hat{x}_k$

USE LATTICE OR BURG TO

UPDATE AR

GIVES $a_{k+1}(z^{-1})$

- GOOD TRANSIENT PROPERTIES
- CONVERGENCE ?

SELF TUNING KALMAN FILTER
JUST SLOT RECURSIVE ESTIMATES
OF F_k, V_k, K_k INTO

$$\hat{x}_{k+1|k} = \hat{F}_k \hat{x}_{k|k-1} + \hat{K}_k \hat{e}_k / \hat{V}_k$$

$$\hat{e}_k = y_k - h' \hat{x}_{k|k-1}$$

WELL IN SF₁, SF₂

$$\hat{K}_k \equiv g_k - \alpha_k; \quad \hat{V}_k \equiv V_k$$

$$\hat{F}_k \equiv \begin{bmatrix} a_{1k} & 0 & \dots \\ \vdots & \ddots & \\ -\alpha_{pk} & 0 & 0 \end{bmatrix}$$

(RECALL COVARIANCE FILTER)

CONVERGENCE $\hat{x}_{k+1|k} - \hat{x}_{k+1|k} \rightarrow 0$

TEDIOUS BUT STRAIGHTFORWARD

SINCE $\hat{K}_k \rightarrow g - \alpha = d - a$

ARMAX

$$(1 + \hat{a}(z^{-1}))y_k = b(z^{-1})u_k + (1 + d(z^{-1}))\epsilon_k$$

USE IVH TO GET \hat{a}, b

POOR TRANSIENT BEHAVIOUR?

SO POWER IN $\hat{u}_k = (1 + \hat{a}_k(z^{-1}))y_k - \hat{b}_k(z^{-1})u_k$

TOO LARGE

SO ADD R STEP \Rightarrow IVHR

REGRESS y_k ON $\psi_k = (-s_{k-1}, u_{k-1})'$

$$s_{1,k} = \hat{a}_k(z^{-1})s_{1,k-1} + \hat{b}_k(z^{-1})u_k$$

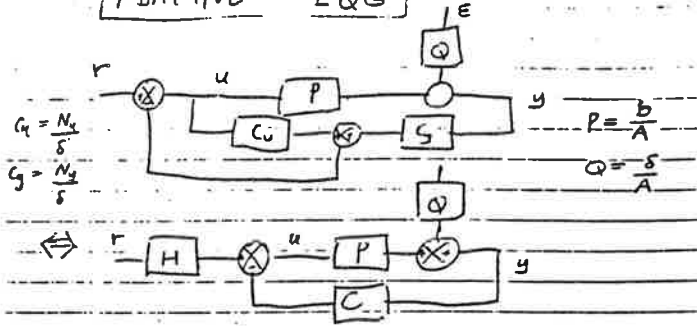
THEN $\hat{w}_k = y_k - \frac{\hat{b}_k(z^{-1})u_k}{1 + \hat{a}_k(z^{-1})}$

HAS BOUNDED POWER

FORM $\hat{u}_k = (1 + \hat{a}_k(z^{-1}))\hat{w}_k$

AND USE $RF_2 \Rightarrow R_2 HF_2$

ADAPTIVE LQG



$$H = \frac{1}{1+G} = \frac{s}{s+N_1} = \frac{s}{-R}$$

$$C = \frac{G}{1+G} = \frac{N_2}{s+N_1} = \frac{s}{R}$$

$$Ay = bu + \delta\epsilon$$

$$Ru = -sy + Hr$$

ESTIMATE A, b, δ FROM FORWARD LOOP BY IVHRSF₂ ($\Rightarrow \delta$ STABLE)

SOLVE $GS = AR + bS$ FOR R_1

$$\& \sigma/|G|^2 = |b|^2 + 2|A|^2$$

DO ONE STEP OF WILSON FOR G

$$|z|=1 \Leftrightarrow z^{-1} \bar{z}$$

$$|D_{t+1}/D_t + \bar{D}_{t+1}/\bar{D}_t - \rho_{t+1}/|\rho_t|^2 + 1 \quad (1)$$

$$|D_{t+1}|^2 = |D_{t+1} - \rho_{t+1}|^2 + \rho_{t+1}^2 \geq \rho_{t+1}^2 \geq \rho_t^2 \geq 0 \quad (2)$$

$$\Rightarrow \text{Re}(D_{t+1}/D_t) = 1 + \frac{\rho_{t+1} \rho_t}{\rho_t^2 |D_t|^2} \geq 1$$

$$\leq 1 + \rho_{t+1}/\rho_t \quad (3)$$

IF $D_t(z)$ HAS NO ZEROS INSIDE $|z|=1$

ROUCHE $\Rightarrow D_{t+1}$ DOES NOT (4)

$\Rightarrow D_{t+1}/D_t$ ANALYTIC IN $|z| \leq 1$

$\Rightarrow \text{Re}(D_{t+1}/D_t)$ IS HARMONIC

\Rightarrow CONSTANT $|z| \leq 1$ OR $\text{MAX} \& \text{MIN. ON } |z|=1$ BUT

SO (3) HOLDS $|z| \leq 1$ (i.e. FOR $z=x$ REAL

ALSO (4) $\Rightarrow D_t$ HAS CONSTANT SIGN (+VE)

$$\Rightarrow D_{t+1}(x)/D_t(x) \leq 1 + \kappa_t$$

$$\alpha_t = \frac{1}{2}(\rho_{t+1}/\rho_t - 1) \rightarrow 0$$

$$\rightarrow D_{t+1}/D_{t+1} \leq D_t/D_t \quad \therefore \pi_t = \pi_t^+(1 + \kappa_t) > 0$$

$$\pi_t^2 = \pi_{t_0}^+ (1 + 2\alpha_t + \alpha_t^2) = \pi_{t_0}^+ \left(\frac{\rho_{t+1}}{\rho_t} + \alpha_t^2 \right)$$

$$= \pi_{t_0}^+ \frac{\rho_{t+1}}{\rho_t} \left(1 + \frac{\alpha_t^2 \rho_t}{\rho_{t+1}} \right) = \frac{\rho_{t+1}}{\rho_t} \pi_{t_0}^+ \left(1 + \frac{\alpha_t^2 \rho_t}{\rho_{t+1}} \right)$$

$\pi_{t_0}^+$ NON DECREASING

BUT $\rho_t \rightarrow \rho \Rightarrow \pi_t \rightarrow \pi$ IF $\pi < \infty$

$$\pi_{t_0}^+ (1 + \alpha_t^2 \rho_t / \rho_{t+1}) < \exp(\sum_{t_0}^+ \alpha_s^2 \rho_s / \rho_{s+1})$$

NEED ONLY $\sum \alpha_s^2 < \infty$

$$\alpha_s^2 = \frac{1}{4} \left(\frac{\rho_{s+1} - \rho_s}{\rho_s} \right)^2 = \frac{(\beta_{s+1} - \rho_s)^2}{4\rho_s^2}$$

$$\rho_{s+1} = \rho_s + (\beta_{s+1} - \rho_s) / (s+1)$$

$$\Rightarrow \alpha_s^2 \leq \frac{1}{2} \frac{\beta_{s+1}^2}{\rho_s^2 s^2} + \frac{1}{2} \frac{1}{s^2}$$

SO $\sum \alpha_s^2 < \infty$ IF $\sum E(\beta_s^2 / s^2) < \infty$

YES IF $E(\beta_s^2) \rightarrow C$

ADAPTIVE SPECTRAL FACTORIZATION

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Despite the great amount of work in the last ten years on the behavior of various recursive parameter estimation and self-tuning control schemes the situation is disappointing. Though a number of global convergence results are available they usually entail restrictions that amount to knowledge of the true system.

Thus, the positive real condition cannot be checked unless the true system parameters are known. Further, monitoring schemes cannot be properly designed unless the true system parameters are known. Also, even for the globally convergent algorithms, various internal filters are not guaranteed to be stable.

In this work some building blocks for algorithm design are suggested. The Levinson, Burg or Lattice algorithm guarantees stability for autoregressive (AR) models. Wilson's (1969) Newton Raphson scheme for spectral factorization guarantees stability of the spectral factor iterate at each iteration. Finally, general regression enjoys a bounded posterior error power property. The use of these building blocks together with the idea of split recursions is illustrated by developing a number of algorithms for ARMA and ARMAX recursive estimation (no iteration is involved).

One of those schemes (called RF_2) is shown to be globally convergent. This scheme is then used to develop a convergent self-tuning Kalman filter. The algorithm is free of the criticisms mentioned above.

Finally, the three tools above are combined to produce a self-tuning LQG controller. It enjoys some stability properties but no convergence proof is available.

Reference

- G. T. Wilson (1969). Factorization of the covariance generating function of a pure moving average process, *SIAM J. Numer. Anal.*, 6, 1-7.

Frequency domain properties of identified transfer function estimates

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Consider the problem of identifying the transfer function of the system

$$y(t) = G_o(q)u(t) + v(t)$$

where y is output, u is input and v is a stationary disturbance with spectrum $\Phi_v(\omega)$.

The input spectrum is supposed to be $\Phi_u(\omega)$. From data up to time N an estimate $\hat{G}_N(e^{i\omega})$ is formed. One method for this is a k -step ahead prediction error method in a given model set

$$y(t) = G(q, \theta)u(t) + H(q, \theta)e(t),$$

$$\hat{G}_N(e^{i\omega}) = G(e^{i\omega}, \hat{\theta}_N)$$

$$\hat{\theta}_N = \underset{\theta}{\operatorname{argmin}} \theta \frac{1}{N} \sum_{t=1}^N \epsilon_F^2(t, \theta)$$

$$\epsilon_F(t, \theta) = L(q) \epsilon(t, \theta)$$

$$\epsilon(t, \theta) = W^k(q, \theta) [y(t) - G(q, \theta) u(t+kT)]$$

$$H(q, \theta) = \bar{H}_k(q, \theta) + q^{-k} \tilde{H}_k(q, \theta) \quad W^k = H_k H^{-1}$$

Several design variables are involved in this method, like k (the prediction horizon), T (the sampling horizon), H (the noise model) and Φ_u .

Then

$$\hat{G}_N(e^{i\omega}) \rightarrow G^*(e^{i\omega}) \quad \text{w.p.1 as } N \rightarrow \infty$$

where $G^*(e^{i\omega})$ is "essentially" determined as the closest function to $G_o(e^{i\omega})$ in the model set, as measured in the weighted L_2 -norm (over the frequencies $-\pi/T \leq \omega \leq \pi/T$) with weighting function

$$Q(\omega) = \Phi_u(\omega) \cdot |L(e^{i\omega})|^2 \cdot |W^k(e^{i\omega})|^2$$

This can be used for a more or less formal discussion of optimal choices of design variables.

References

- L. Ljung: Estimation of transfer functions. Report LiTH
- L. Ljung: Asymptotic variance expressions for identified transfer function estimates. Report LiTH
- B. Wahlberg & L. Ljung: Design variables for bias distribution of identified transfer functions. Report LiTH

FREQUENCY DOMAIN PROPERTIES OF IDENTIFIED TRANSFER FUNCTION ESTIMATES

Lennart Ljung & Bo Wahlberg

PROBLEM:

True system:

$$y(t) = G(q) u(t) + v(t)$$

Observed data:

$$Z^N: u(1), y(1), \dots, u(N), y(N)$$

Model: $\hat{G}_N(q)$

WHAT CAN WE SAY ABOUT

$$\hat{G}_N(e^{j\omega}) - G_0(e^{j\omega})$$

AS A FUNCTION OF ω ?

3

1. BIAS & RANDOM ERRORS

Let

$$G^*(e^{j\omega}) = E \hat{G}_N(e^{j\omega}) \quad (E \text{ w.r.t. } \{v(t)\}_1^N)$$

$$G^*(e^{j\omega}) - G_0(e^{j\omega}) \quad \text{BIAS error}$$

$$\hat{G}_N(e^{j\omega}) - G^*(e^{j\omega}) \quad \text{RANDOM error}$$

$$\hat{G}_N - G_0 = \underbrace{\hat{G}_N - G^*(e^{j\omega})}_{\text{RANDOM error}} + \underbrace{G^*(e^{j\omega}) - G_0(e^{j\omega})}_{\text{BIAS error}}$$

"Traditional" analysis neglects ↗

Typically: $\sqrt{N}[\hat{G}_N(e^{j\omega}) - G^*(e^{j\omega})] \in \text{AsN}(0, P(\omega))$

Expressions for P_1, \dots

Here, we shall concentrate on bias

1. Bias error & random error
2. k-step ahead prediction error methods
3. conventional analysis design issues
4. A frequency domain expression for the limiting estimate
5. Choice of u
 L
 k
 T
6. Conclusions

4

2. PREDICTION ERROR METHODS

Choose:

- ① T sampling interval
- ② $\Phi_u(\omega)$ input spectrum
- ③ Model set $\mathcal{G} = \{G(q, \theta)\}$ $\mathbb{R}^n = G_0 + H_0 \mathbb{I}$
- ④ Noise model set $\mathcal{H} = \{H(q, \theta)\}$
- ⑤ Prediction horizon k
 $H = \bar{H}_k + q^{-k} \tilde{H}_k \quad W^k = \bar{H}_k H^{-1}$
 $\hat{y}(t+kT|t, \theta) = W^k(q, \theta) G(q, \theta) u(t+kT) + (1-W^k) \hat{y}(t|t, \theta)$
 $E(t+kT|t, \theta) = W^k(q, \theta) [y(t) - G(q, \theta) u(t+kT)]$
 $[k=1 \Rightarrow W^k = H^{-1}]$
- ⑥ Filter $L(q)$
 $E_F(t+kT|t, \theta) = L(q) E(t+kT|t, \theta)$

Then

$$\hat{G}_N = \underset{\theta}{\text{argmin}} \frac{1}{N} \sum_{t=1}^N E_F^2(t+kT|t, \theta)$$

4. THE LIMITING ESTIMATE

$$\hat{\Theta}_N \rightarrow \Theta^* = \arg \min E E^*(t+h, t, \Theta) \quad \text{w.p.1}$$

$$\Theta^* = \arg \min \int_{-\pi}^{\pi} [|G_0 - G(\theta)|^2 \Phi_u + \Phi_v] \cdot |L|^2 |W_k(\theta)|^2 d\omega$$

Fixed noise model $H(\theta) = H^* (W_k(\theta) = W_k^*)$

$$\hat{\Theta}^* = \arg \min \int_{-\pi}^{\pi} |G_0 - G(\theta)|^2 Q^* d\omega$$

$$Q^*(\omega) = \Phi_u(\omega) |L(e^{j\omega})|^2 |W_k^*(\omega)|^2$$

Independently parametrized: $G(\theta) = G(\rho)$; $H(\theta) = H(\rho)$

$$\hat{\rho}^* = \arg \min \int_{-\pi}^{\pi} |G_0 - G(\rho)|^2 Q(\rho^*) d\omega$$

$$Q(\rho) = \Phi_u(\omega) |L(e^{j\omega})|^2 |W_k(\rho, \omega)|^2$$

$$\rho^* = \arg \min \int_{-\pi}^{\pi} [|G_0 - G(\rho^*)|^2 \Phi_u + \Phi_v] |L(e^{j\omega})|^2 |W_k(\rho^*, \omega)|^2 d\omega$$

General case:

$$\Theta^* = \arg \min \left[\int_{-\pi}^{\pi} |G_0 - G(\theta)|^2 Q(\theta^*) d\omega + \int_{-\pi}^{\pi} [|G_0 - G(\theta^*)|^2 \Phi_u + \Phi_v] |L(e^{j\omega})|^2 |W_k(\theta^*, \omega)|^2 d\omega \right]$$

7

3. Design issues

$\hat{G}_N(e^{j\omega})$ depends on all the listed Θ choices.

What are 'optimal' choices of

- noise model
- prediction horizon
- prefilter
- input spectrum
- sampling interval

Traditional analysis: "optimal" = min. var.

CONCLUSION:

The weighting function

$$Q(\omega, \Theta^*) = \Phi_u(\omega) |L(e^{j\omega})|^2 |W_k(\Theta^*, \omega)|^2$$

determines, entirely or partly, the bias distribution

Formal design problem:

design variables:

$$\mathcal{D} = \{ \Phi_u(\omega), k, L(\rho), H(\rho) \}$$

$$\Theta^* = \Theta^*(\mathcal{D})$$

design criterion

$$J(\mathcal{D}) = \int_{-\pi}^{\pi} |G(\Theta^*(\mathcal{D})) - G_0|^2 G(\omega) d\omega$$

$$\min_{\mathcal{D} \in \mathcal{D}} J(\mathcal{D})$$

recall:

$$\Theta^*(\omega) = \arg \min \int_{-\pi}^{\pi} |G(\theta) - G_0|^2 \underbrace{\Phi_u(\omega) \cdot |L|^2 |W_k|}_{\mathcal{D}} d\omega$$

SOLUTION: $\mathcal{D}_{opt} = \Phi_u(\omega) |L(e^{j\omega})|^2 |W_k(e^{j\omega})|^2 \propto C$

System:

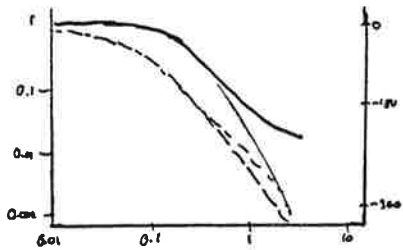
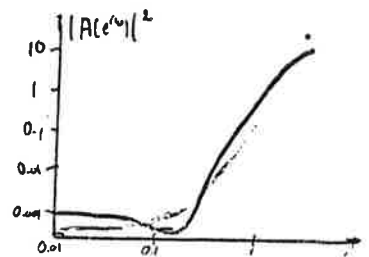
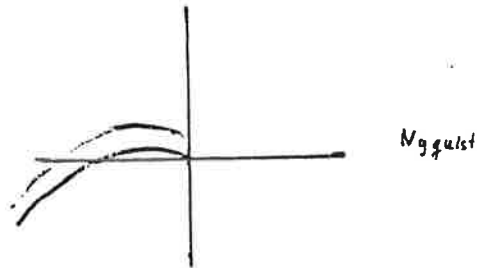
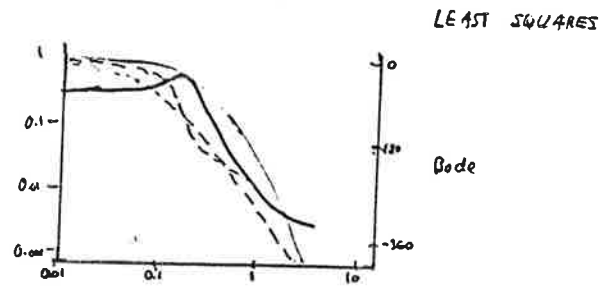
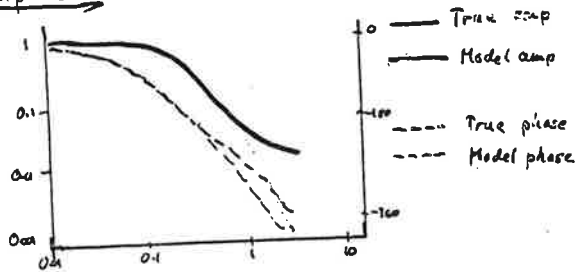
$$y(t) = 2.14 y(t-1) + .555 y(t-2) - 0.44 y(t-3) + 0.042 y(t-4) + 0.01 u(t-3) + 0.0074 u(t-1) + 0.000921 u(t-4) - 0.000176 u(t-5)$$

Model:

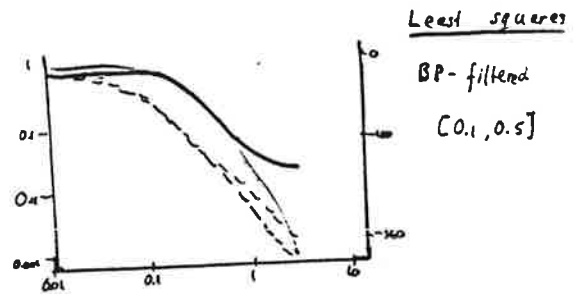
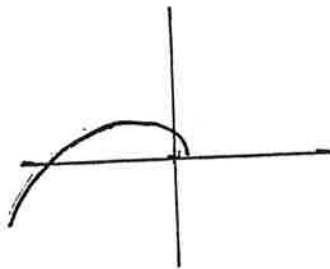
$$y(t) + a_1 y(t-1) + a_2 y(t-2) = b_1 u(t-1) + b_2 u(t-2)$$

Input: PRBS - white noise

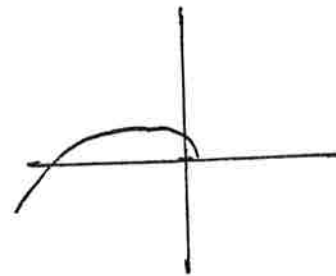
Output error: $H^2 = 1$

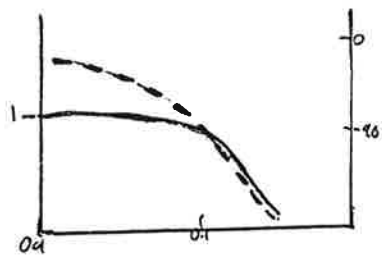


LEAST SQUARES
LP filtered
[0, 0.1]

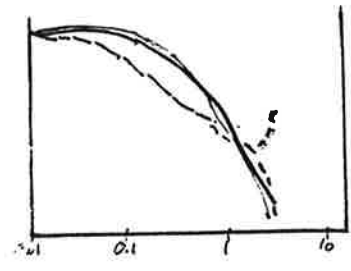


Least squares
BP-filtered
[0.1, 0.5]

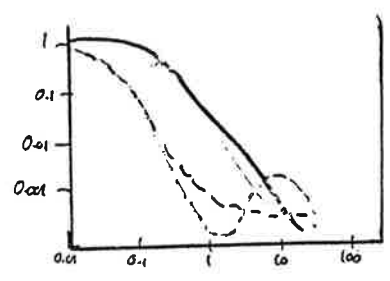




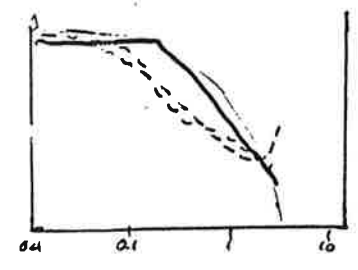
$T=10$



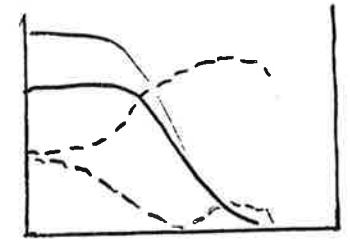
$T=1$
slow input



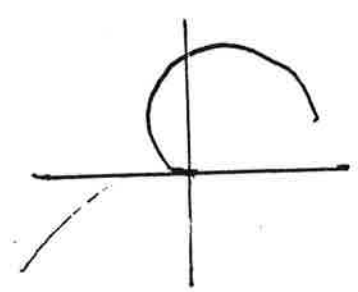
$T=0.1$
slow input



$T=1$
fast input



$T=0.1$
fast input



CONCLUSION

The bias distribution is determined by a frequency domain weighting function that is influenced by the noise model, the prediction horizon, the prefilter, the input spectrum and the sampling interval

Multi-armed bandits

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Contents

- 1) Bandit process
- 2) Multiarmed
- 3) Index
- 4) Adaption

1. A bandit process is $\{(x(1), F(1)); (x(2), F(2))\dots\}$, where
 $x(s)$ = reward on the s th continuation
 $F(s)$ = σ -field of information after $s-1$ plays.

Ex Coin toss

$x(s) \in \{0,1\}$. $F(0)$ = prior information about success
 $F(s) = F(0) \cup \{x(0)\dots x(s-1)\}$

2. A multiarmed bandit is defined according to

$$\{x^i(s), F^i(s)\} \quad i = 1, \dots, n \quad F^i(\infty) \perp\!\!\!\perp F^j(\infty) \quad i \neq j$$

$$t - 1 = t^1 + \dots + t^n ; F^1(t^1+1) \cup \dots \cup F^n(t^n+1) = F(t)$$

The control $u(t) \in \{1\dots n\}$

$u(t)$ is the decision which bandit that shall continue.

$$u(t) = i \longrightarrow \begin{cases} R(t) & = x^i(t^i+1) \\ F(t+1) & = F(t) \cup F^i(t^i+1) \end{cases}$$

The control $u(t)$ has to maximize the total reward

$$V_\beta(\Pi) = \text{Max } E \sum_{t=1}^{\infty} \beta^t R(t) \quad 0 < \beta < 1$$

β close to zero \rightarrow we want to maximize the reward in the first play(s).

β close to unity \rightarrow all rewards are equally valuable

There is a conflict between getting immediate rewards and learning more about the other bandits.

3. Index of a bandit process $\{x(s), F(s)\}$

$$\nu_{\beta}(s) = \text{Max}_{\tau > 1} \frac{E \left\{ \sum_{t=s}^{\tau-1} \beta^t x(t) \mid F(s) \right\}}{E \left\{ \sum_{t=s}^{\tau-1} \beta^t \mid F(s) \right\}} = \frac{\text{Acc. reward}}{\text{"Acc. time"}}$$

τ ranges over the stopping times of $F(s)$. The best policy is to continue the bandit with the largest index (multiarmed case).

Notice that the calculation of ν_{β} for a bandit involves only the bandit itself.

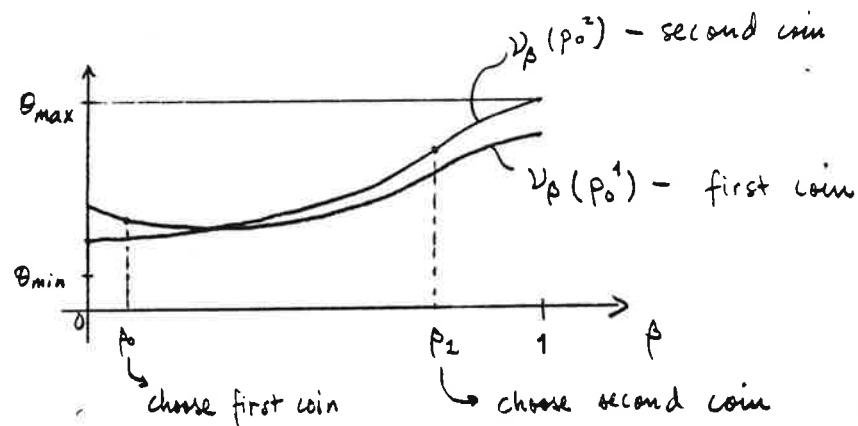
Two coins θ_1 and θ_2

Fact about index $\nu_{\beta}(\bar{s}) = \nu_{\beta}(p)$ (distribution of success probability).

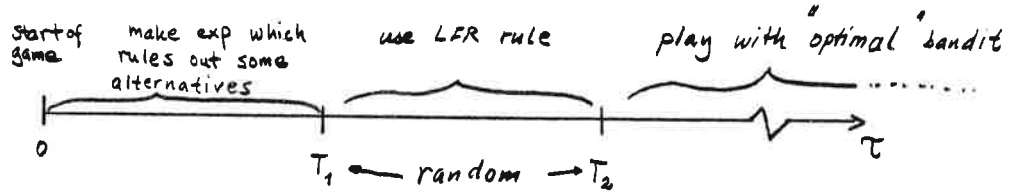
1) Expected reward just now

$$\nu_0(p) = \bar{p} = \text{exp of } \theta$$

2) $\nu_1(p) = \text{Max } \theta$ (the most favourable possible event)
 $p(\theta) > 0$



Optimal solution for $\beta = 1, r \rightarrow \infty$
 (infinite horizon problem) [Kelly Am. stat. 1981]



LFR = least failure rule: Try the one that currently looks best. As soon as it fails (no reward), choose another one.

4.

The procedure above constitutes a kind of adaptation.
 Unsolved problems: Transient behavior
 Length of the first two phases.

References

- F.P. Kelly: Multi-armed bandits with discount factor near one: the Bernoulli case, Ann. Statist 9, 1981, 987-1001.
- P. Varaya, J. Walsand and C Bnykkoc: Extensions of the multi-armed bandit problem: the discounter case, IEEE AC Trans, 1985, to appear.
- J. C. Gittins: Bandit processes and dynamic allocation indices, J. Royal Stat. Soc 41, 1979, 148-177.

On self-tuning to the optimal controller

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Summary

The question of whether some adaptive controllers automatically tune themselves to the optimal control law is examined.

First we consider an adaptive control law consisting of a Stochastic Approximation (or Stochastic Gradient) parameter estimator followed by a minimum variance control law. Under some conditions it is shown that, see [1,2].

- i) the parameter estimates converge to a random multiple of the true parameter,
- ii) and so the adaptive control law converges to the true minimum variance control law,
- iii) even though the standard persistency of excitation condition is not satisfied.

Next we consider the same parameter estimator followed by a linear quadratic control law. Ljung's O.D.E.'s are examined to show that though the parameter estimates may converge, they will not generally converge to an optimal control law.

In fact, the result for the case of the minimum variance control law rests on a fortuitous mathematical coincidence, and it is unlikely that self-tuning to the optimal will occur for general cost criteria, see [3].

References

- A. Becker, P. R. Kumar and C. Z. Wei: Adaptive Control with the Stochastic Approximation Algorithm: Geometry and Convergence, to appear in IEEE Transactions on Automatic Control, March 1985.
- P. R. Kumar: A Survey of Some Results in Stochastic Adaptive Control, March 1985.
- W. Lin, P. R. Kumar and T. I. Seidman: Will the Self-tuning Approach Work for General Cost Criteria?

SELF-TUNNING

QUESTION : DOES SELF-TUNNING TAKE PLACE?
 YES : MINIMUM VARIANCE CONTROLLER
 NO : OPTIMAL SELF-TUNNING

QUESTION : Does Self-Tuning Take Place?

Will the adaptive controller converge to the optimal controller?

Yes : MINIMUM VARIANCE CONTROLLER
NO : OPTIMAL CONTROLLER

No : OPTIMAL CONTROLLER
 GENERAL COST CRITERION

Speculative : Self-Tuning Result is not possible in general by straightforward schemes.

SYSTEM

$$y_{k+1} = a_1 y_k + a_2 y_{k-1} + b_1 u_k + b_2 u_{k-1} + w_{k+1}$$

MINIMUM VARIANCE CONTROL LAW

$$u_k = -\frac{1}{b_1} \left[(a_1 + a_2) y_k + (a_2 + b_1) y_{k-1} + b_2 y_{k-2} + b_1 u_{k-1} + b_2 u_{k-2} \right]$$

MORE CONVENIENTLY

$$y_k = \begin{pmatrix} y_k & y_{k-1} & y_{k-2} & u_{k-1} & u_{k-2} \end{pmatrix}^T$$

$$u_k = \begin{pmatrix} u_k & u_{k-1} & u_{k-2} \end{pmatrix}^T$$

$$y_{k+1} = \Phi_k^T \theta + w_{k+1}$$

ADAPTIVE SCHEME

ESTIMATE : $\hat{\theta}_k$

CONTROL : $\Phi_k^T \hat{\theta}_k = 0$

Notes: One degree of freedom

It is enough if $\hat{\theta}_k \rightarrow x \in \mathbb{R}^5$

STOCHASTIC APPROXIMATION

OR STOCHASTIC GRADIENT ALGORITHM

$$\hat{\theta}_{N+1} = \hat{\theta}_N + \frac{\gamma}{N} (y_{N+1} - \Phi_N^T \hat{\theta}_N)$$

$$y_{N+1} = \Phi_N^T \theta + w_{N+1}$$

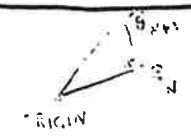
More convenient

$$\hat{\theta}_{N+1} = \hat{\theta}_N + \frac{\gamma}{N} (y_{N+1} - \Phi_N^T \hat{\theta}_N) \tau$$

τ chosen such that $\Phi_N^T \hat{\theta}_N = 0$

What does such an algorithm do?

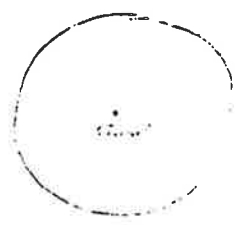
- $\hat{\theta}_{N+1} = \hat{\theta}_N + \tau (y_{N+1} - \Phi_N^T \hat{\theta}_N)$
- $\hat{\theta}_{N+1} - \hat{\theta}_N$ parallel to Φ_N
- τ perpendicular to $\hat{\theta}_N$
- Hence $\hat{\theta}_{N+1} - \hat{\theta}_N \perp \hat{\theta}_N$



• $\hat{\theta}_N$ converges to θ^* if $\|\hat{\theta}_N\| \geq \|\theta^*\|$

• Hence $\lim \|\hat{\theta}_N\| \geq \|\theta^*\|$

• $\hat{\theta}_N$ converges to random vector θ^* if $\|\hat{\theta}_N\| \geq \|\theta^*\|$



• S.F. $\|\hat{\theta}_N - \theta^*\| \rightarrow 0$

• S.F. $\|\hat{\theta}_N\| \rightarrow \|\theta^*\|$

• $\|\hat{\theta}_N - \theta^*\|$ converges

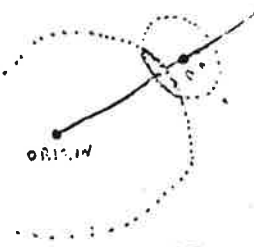


• $\hat{\theta}_N$ converges to random vector θ^* if $\|\hat{\theta}_N\| \geq \|\theta^*\|$

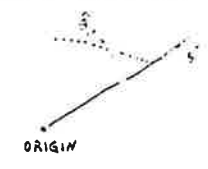
ORIGIN

• $\hat{\theta}_N$ converges to hypersphere

• Line and hypersphere do not intersect except if hypersphere = point



• Control system $\hat{\theta}_N \rightarrow \text{Line}$



• Proof that $\hat{\theta}_N \rightarrow \text{Line}$

$$\hat{\theta}_N = \frac{1}{N} \sum_{i=1}^N \theta_i$$

$$\theta_i = -\frac{1}{\hat{\theta}_i} (\hat{\theta}_i \theta_i + \hat{\theta}_i \theta_{i-1})$$

$$\hat{\theta}_N = \left(\frac{1}{N} \sum_{i=1}^N \frac{1}{\hat{\theta}_i} \right) \theta_1 + \left(\frac{1}{N} \sum_{i=1}^N \frac{1}{\hat{\theta}_i} \right) \theta_{i-1}$$

• Optimality: $\lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N \frac{1}{\hat{\theta}_i} = \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N \frac{1}{\hat{\theta}_i}$

• Analysis

$$\lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N \left\{ \left(\frac{1}{\hat{\theta}_i} \right) \theta_1 + \left(\frac{1}{\hat{\theta}_i} \right) \theta_{i-1} \right\}$$

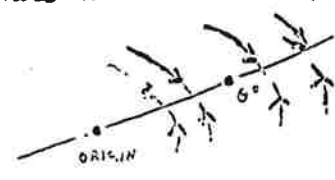
• Local Convergence Theorem + Analysis

$$\lim_{N \rightarrow \infty} \frac{1}{N} \sum_{i=1}^N \left(\frac{1}{\hat{\theta}_i} \right) + \left(\frac{1}{\hat{\theta}_i} \right) \theta_{i-1} = 0$$

$$\left(\frac{1}{\hat{\theta}_i} \right)_{N \rightarrow \infty} \rightarrow \frac{1}{\theta_0} \quad \text{and} \quad \left(\frac{1}{\hat{\theta}_i} \right)_{N \rightarrow \infty} \rightarrow \frac{1}{\theta_1}$$

Conclusions

• $\hat{\theta}_N \rightarrow \kappa \theta^*$ where $\kappa = \text{random scalar}$.



• $\hat{\theta}_N \not\rightarrow \theta^*$: Does not converge to true parameter.

• Regulator converges to optimal

SELF-TUNING

$$\|\hat{\theta}_N\| \geq \|\theta^*\|$$

$$\lim_{N \rightarrow \infty} \|\hat{\theta}_N\| \geq \|\theta^*\| = \|\theta^*\|$$

So convergence impossible to θ^*

OPEN QUESTIONS

• Proof of self-tuning law
i) L.S scheme ii) Delay iii) Tracking

• Rate of convergence
Control Limit Theorem

• Adaptive MIMO, P, PI, PD, PID

• Robustness

• Self-tuning of θ^*

• Self-tuning of θ^*

• More sophisticated control laws.

STATUS

• Much progress since 1973
• Much remains to be done.

LINEAR QUADRATIC CONTROL LAW

$$x_{k+1} = ax_k + bu_k + w_{k+1}$$

$$Cost = \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{k=0}^N y_k^2 + \rho u_k^2$$

$$u_k = K(a, b) y_k \quad \text{is optimal}$$

ADAPTIVE CONTROL SCHEME

$$\hat{x}_{k+1} = \begin{pmatrix} \hat{x}_k \\ \hat{y}_k \end{pmatrix} + \frac{1}{N} (y_k - \hat{y}_k) \begin{pmatrix} 1 \\ -\hat{a}_k - \hat{b}_k K(\hat{a}_k, \hat{b}_k) \end{pmatrix}$$

$$u_k = K(\hat{a}_k, \hat{b}_k) y_k$$

REWRITE

$$\begin{pmatrix} \hat{x}_{k+1} \\ \hat{y}_{k+1} \end{pmatrix} = \begin{pmatrix} \hat{x}_k \\ \hat{y}_k \end{pmatrix} + \frac{1}{N} (y_k - \hat{y}_k) \begin{pmatrix} 1 \\ -\hat{a}_k - \hat{b}_k K(\hat{a}_k, \hat{b}_k) \end{pmatrix}$$

LYUNG'S ODE

$$\frac{d}{dt} \begin{pmatrix} a(t) \\ b(t) \end{pmatrix} = c(a, b) \begin{pmatrix} 1 \\ K(a, b) \end{pmatrix}$$

$$\frac{db}{da} = K(a, b)$$



... all these curves will converge to the same point...

Why did Min Variance Case Work?

Mathematical Coincidence

$$K(a, b) = -\frac{a}{b}$$

$$K(a^*, b^*) = -\frac{a^*}{b^*} K(a^*, b^*) = 0$$

$$\Rightarrow K(a^*, b^*) = -\frac{a^*}{b^*} = K(a^*, b^*)$$

... all these trajectories are attracted...

$$\begin{pmatrix} \hat{x}_{k+1} \\ \hat{y}_{k+1} \end{pmatrix} = \begin{pmatrix} \hat{x}_k + \frac{1}{N} (y_k - \hat{y}_k) \\ \hat{y}_k + \frac{1}{N} (y_k - \hat{y}_k) K(\hat{a}_k, \hat{b}_k) \end{pmatrix}$$

$$\{ [a + bK(\hat{a}_N, \hat{b}_N)] - [\hat{a}_N + \hat{b}_N K(\hat{a}_N, \hat{b}_N)] \} y_N + w_{N+1}$$

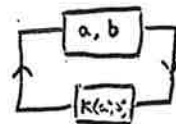
$$\frac{1}{N} \begin{pmatrix} \hat{x}_{N+1} \\ \hat{y}_{N+1} \end{pmatrix} = \frac{1}{N} \begin{pmatrix} \hat{x}_N \\ \hat{y}_N \end{pmatrix} + \frac{1}{N} \begin{pmatrix} 1 \\ -\hat{a}_N - \hat{b}_N K(\hat{a}_N, \hat{b}_N) \end{pmatrix} (y_N - \hat{y}_N)$$

EQUILIBRIUM POINTS OF O.D.E.

$$(a^*, b^*) : a + bK(a^*, b^*) = a^* + b^* K(a^*, b^*)$$

CLOSED LOOP IDENTIFICATION PROBLEM

Can only identify $a + bK(a^*, b^*)$



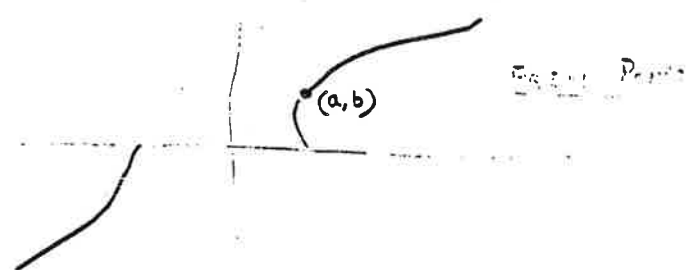
So, (\hat{a}_N, \hat{b}_N) MAY converge to such (a^*, b^*)

Question: We want self-tuning. \exists

$$\text{Does } a + bK(a^*, b^*) = a^* + b^* K(a^*, b^*) \Rightarrow K(a^*, b^*) = \dots$$

$$\text{No: } a + bK(a^*, b^*) = a^* + b^* K(a^*, b^*)$$

implies $K(a^*, b^*) = K(a, b)$ only if $(a, b) = (a^*, b^*)$



HOPE: IF THERE ARE OTHER UNDESIREL EQUILIBRIUM POINTS, ARE THEY REPELLERS?

$$\frac{d}{dt} \begin{pmatrix} a(t) \\ b(t) \end{pmatrix} = c(a, b) \begin{pmatrix} 1 \\ K(a, b) \end{pmatrix}$$

... all these trajectories will converge to the same point...

INTEGRAL CURVES!

$$\frac{db(t)}{da(t)} = K(a(t), b(t))$$

NOTE: VALID NO MATTER WHAT $K(\cdot, \cdot)$ is

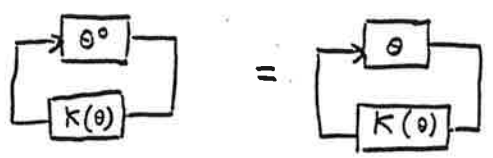
ENTRATIVE CONCLUSIONS

Self Tuning to optimal is necessary for
 Minimum Variance Control Law
 Because of MATHEMATICAL COINCIDENCE
 Self Tuning to optimal is also possible
 for Minimum Variance Control Law
 Self Tuning to Pole Placement is possible

Closed Loop Pole = $a + bK(a^*, b^*)$
 = $a^* + b^* K(a^*, b^*)$
 = Desired Pole

Self Tuning to STIC Control Law
 $a^* + b^* K(a^*, b^*)$ stable since $K(a^*, b^*)$ chosen so
 \Downarrow
 $a + bK(a^*, b^*) = a^* + b^* K(a^*, b^*)$ also stable.
 Approx. Transfer Function $T(s, \theta) \approx D(s, \theta)$
 Hence Bias Parameter Estimates. See MP literature.

What properties does D have?



$T(s, \theta) = \frac{D(s, \theta)}{K(s, \theta)} = \frac{D(s, \theta)}{D(s, \theta)}$
 $\Rightarrow T(s, \theta) = 1$
 $\Rightarrow \frac{D(s, \theta)}{K(s, \theta)} = 1$
 $\Rightarrow D(s, \theta) = K(s, \theta)$

$\hat{\theta}_N$: minimize $\frac{1}{N} \log J(\theta) + \sum_{t=1}^N \{y(t) - \hat{y}(t|\theta)\}^2$
 Such solutions do not depend on the
 plant to be identified.

FUNDAMENTAL CLOSED LOOP IDENTIFICATION PROBLEM

Estimate (SA, LS, ML)
 $\hat{y}_t = K(z^{-1})u_t$
 $T(z^{-1}) = \frac{A(z^{-1}) + B(z^{-1})K(z^{-1})}{A(z^{-1}) + B(z^{-1})K(z^{-1})}$
 $= 1$

Question: How do you estimate this?

PROPERTY

- Proof of Self-Tuning to Minimum Variance Controller
- Self Tuning probably not possible for any other cost criteria
- Method to obtain self-tuning sorts of results in general

NOTE

- Pole placement, self-tuning, etc.
- Time-varying systems, self-tuning, etc.
- Control (self-tuning)

A comparison of some control strategies for systems with fast parameter changes

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This seminar is concerned with adaptive control of systems with abrupt changes in the parameters. First an algorithm for recursive identification of the parameters in a special model-class, suitable for modeling sudden time variations, is presented. Then it is shown how this identification procedure can be used for adaptive control. To illustrate the algorithm and also to discuss some points on adaptive control a comparison study is then referenced.

The basic idea behind the algorithm is to use several parameter sets to model the system. The different parameter vectors correspond to different typical modes of the system. By combining estimation and detection techniques it is possible to estimate the different parameter vectors describing the system.

The purpose of the comparison study was to compare some strategies for adaptive control. Among the "competitors" were for instance a self-tuning regulator (LS with forgetting factor combined with a pole-placement procedure), a time-invariant robust regulator, the regulator mentioned above and an algorithm based on an identification procedure called AFMM (adaptive forgetting through multiple models). A conclusion from the tests was that it can be useful to design regulators which saves information for later use.

References

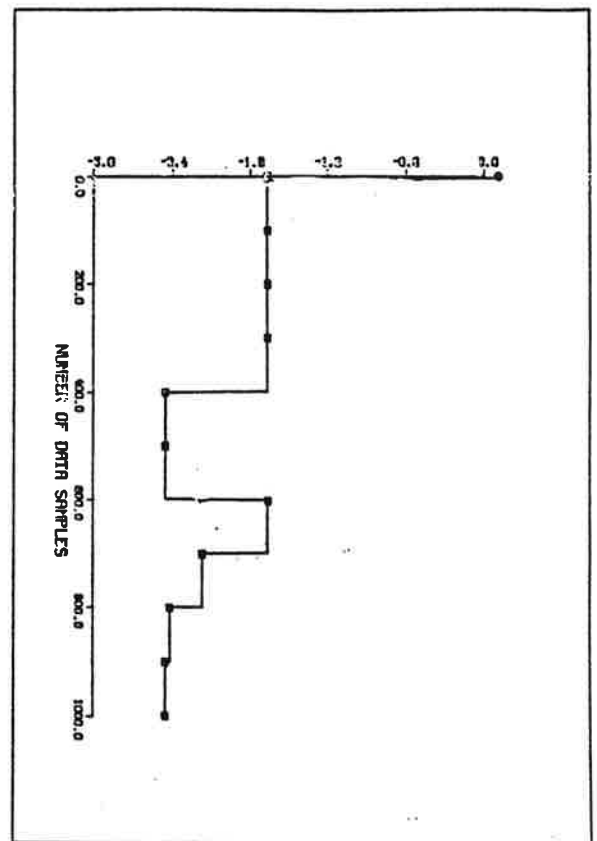
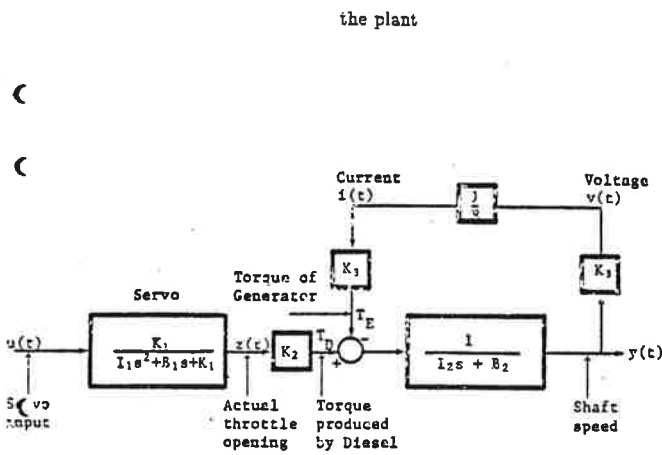
- Millnert, M. (1982): Identification and Control of Systems Subject to Abrupt Changes, Linköping Studies in Science and Technology, Dissertation. No. 82
Millnert, M. (1983): Control Strategies for Systems with Abruptly Changing Parameters, INTERNAL REPORT, LiTH-I-ISY-0634.
Andersson, P. (1983): Adaptive Forgetting in Recursive Identification through Multiple Models, INTERNAL REPORT, LiTH-ISY-I-0638.

A COMPARISON OF SOME CONTROL STRATEGIES FOR SYSTEMS WITH FAST PARAMETER CHANGES

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 Sweden

A contest between some strategies for adaptive control

THE CONTEST



THE COMPETITORS

- 1 A self tuner (LC + pole-placement)
- 2 A constant regulator minimizing a variance-type criterion
- 3 A regulator minimizing the criterion $E\{(y-r)^2 + \lambda u^2\}$
- 4 The MR algorithm
- 5 The AFMM algorithm (forgetting through multiple models)

$$\theta(t+1) = \theta(t) + w(t)$$

$$y(t) = \phi^T(t) \theta(t) + e(t)$$

$$w(t) = \begin{cases} V(t) & \text{w.p. } q \\ 0 & \text{w.p. } 1-q \end{cases}$$

$$\text{COV } V(t) = R_1$$

5

$P(\theta_i(t) | Y^t) =$ Gaussian sum based on \bar{E}_i^t $i=1 \dots m$

$\bar{\theta}_i \leftarrow$ least squares

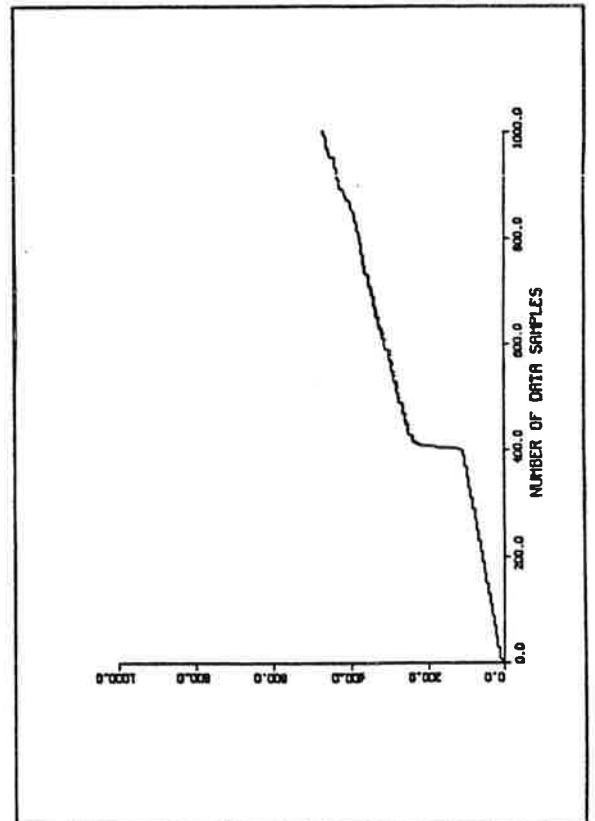
At each time

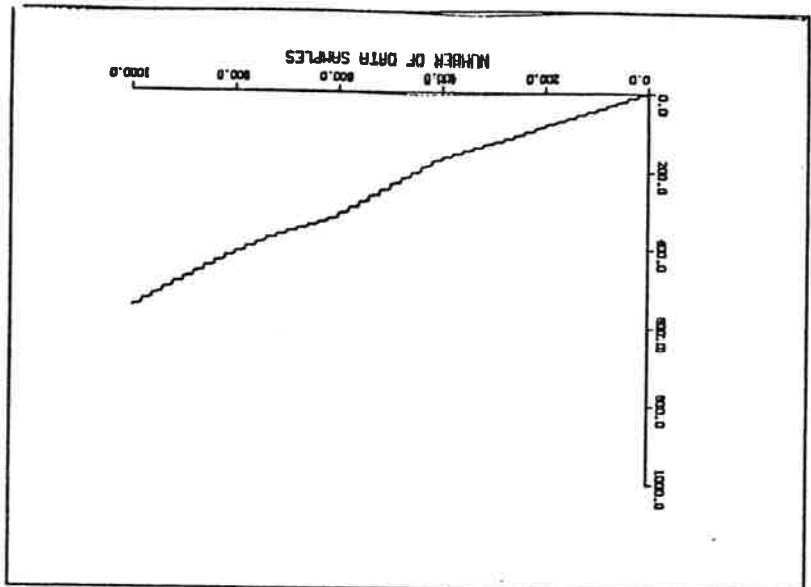
~~$\bar{\theta}_i(t)$~~ with smallest prob.

$\bar{\theta}_i(t)$ with largest prob.

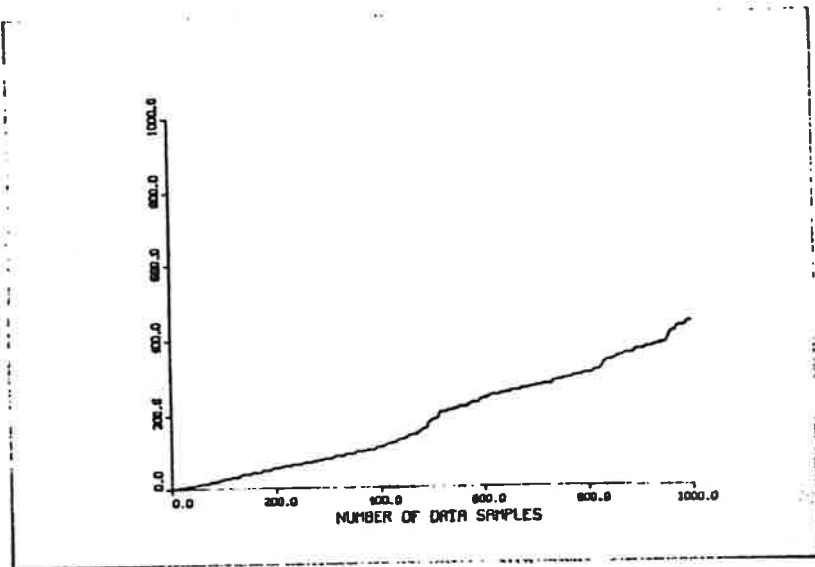
\downarrow
 $\bar{\theta}_i(t)$ "Test pilot"
 alert $P_i = R_1$

$\Sigma(t-y)$

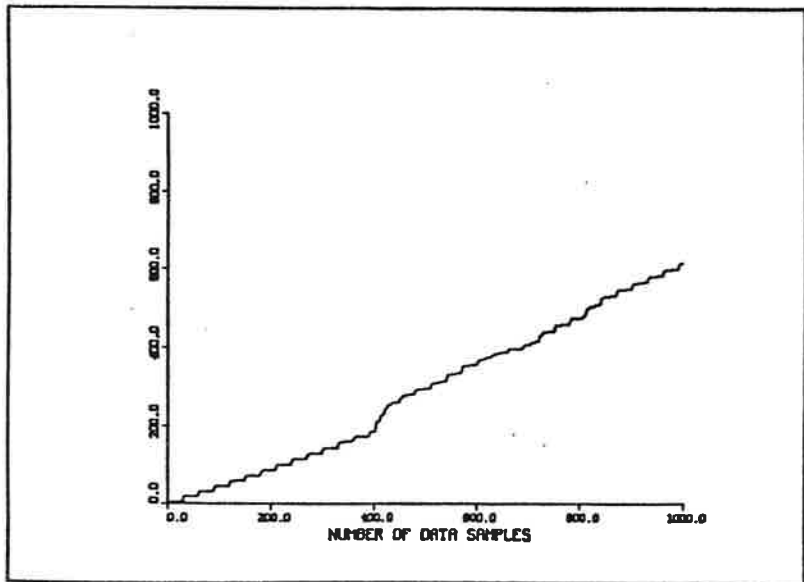




KN 302



13



Recursive estimation of slowly time-varying parameters

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Abstract

The problem to extend traditional algorithms for estimation of constant parameters, such as the least squares algorithm, to capture even the case of time-variable parameters has become important because of their use in adaptive control. Several ad hoc methods have been proposed to handle slowly time-varying parameters. Previous methods, such as the use of a forgetting factor, are here discussed from an information handling point of view. A new method is presented, which is based on the idea to retain a constant amount of information in the estimator. The method is shown to avoid well-known problems associated with other, more heuristic schemes. Analysis as well as simulation experiments are presented.

References

- Hägglund, T. The Problem of Forgetting Old Data in Recursive Estimation. Proceedings IFAC Workshop on Adaptive Systems in Control and Signal Processing. San Francisco 1983.
- Hägglund, T. New Estimation Techniques For Adaptive Control. PhD-thesis. Report TFRT-1025. Department of Automatic Control, Lund Institute of Technology, Lund Sweden. 1983.

RECURSIVE ESTIMATION OF

SLOWLY TIME-VARYING PARAMETERS

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DEPT. OF AUTOMATIC CONTROL

LUND SWEDEN

LS ESTIMATION

$$\text{MINIMIZE } \sum_{i=1}^t \frac{1}{\omega(t,i)} [Y(i) - \Theta(t)^T \varphi(i)]^2$$

PROBLEM: HOW TO CHOOSE ω

$$\begin{aligned} Y(i) &= \Theta(i-1)^T \varphi(i) + e_n(i) = \\ &= \Theta(t-1)^T \varphi(i) + [\Theta(i-1) - \Theta(t-1)]^T \varphi(i) + e_n(i) = \\ &= \Theta(t-1)^T \varphi(i) + \underbrace{e_m(t,i)}_{\text{model error}} + \underbrace{e_n(i)}_{\text{noise}} \quad i \leq t \end{aligned}$$

$$\sigma(t,i)^2 = \sigma_m(t,i)^2 + \sigma_n(i)^2$$

TRY TO CHOOSE $\omega = \sigma^2$

TIME-VARYING PARAMETERS

PROCESS MODEL:

$$Y(t) = \Theta(t-1)^T \varphi(t) + e_n(t)$$

IF THE PARAMETERS TO BE ESTIMATED OR THE NOISE LEVEL VARY, THEY VARY SLOWLY AND/OR SELDOM COMPARED WITH THE TIME CONSTANTS OF THE SYSTEM.

TWO CASES:

1. LARGE PARAMETER CHANGES
2. SLOW PARAMETER CHANGES

EXAMPLE: CONSTANT PARAMETERS AND CONSTANT NOISE LEVEL

$$e_m(t,i) = [\Theta(i-1) - \Theta(t-1)]^T \varphi(i) = 0$$

$$\Rightarrow \sigma_m(t,i) = 0$$

$$\Rightarrow \sigma(t,i)^2 = \sigma_n(t,i)^2 = \sigma^2$$

ALL MEASUREMENTS HAVE THE SAME WEIGHT

EXAMPLE: EXPONENTIALLY INCREASING MODEL ERROR VARIANCE AND CONSTANT NOISE LEVEL

$$\sigma_m(t,i)^2 = \left[\left(\frac{1}{\lambda} \right)^{t-i} - 1 \right] \sigma^2 \quad \sigma_n(i)^2 = \sigma^2$$

$$\Rightarrow \sigma(t,i)^2 = \left(\frac{1}{\lambda} \right)^{t-i} \sigma^2$$

CONSTANT FORGETTING FACTOR λ

SLOW PARAMETER CHANGES

DISCOUNT PAST DATA IN SUCH A WAY THAT A CONSTANT DESIRED AMOUNT OF INFORMATION IS RETAINED, IF THE PARAMETERS ARE CONSTANT.

INFORMATION: P^{-1}

Goal: $P \rightarrow a \cdot I$

$$P_k' = P_{k-1} + \frac{1}{V_k} \psi_k \psi_k^T - \alpha_k \psi_k \psi_k^T \quad \alpha_k \geq 0$$

$$P_k = P_{k-1} - \frac{P_{k-1} \psi_k \psi_k^T P_{k-1}}{(V_k - \alpha_k) + \psi_k^T P_{k-1} \psi_k}$$

COMPARE:

$$P_k' = P_{k-1}' + \frac{1}{V} \psi_k \psi_k^T - (1-\lambda) P_{k-1}'$$

CONVERGENCE

IF a NOT TOO LARGE (cf $\lambda < 0$)

THEN

$$W(k) = \sum_1^n [\lambda_i(k) - a]^2 \quad \lambda_i(k) = \text{eig}(P_k)$$

IS DECREASING.

$$W(k) \rightarrow 0 \Rightarrow P(k) \rightarrow a \cdot I$$

EXCITATION CONDITION $\Rightarrow W(k) \rightarrow 0$

CHOICE OF $\alpha(k)$

STATIONARITY: $\alpha_k = \frac{1}{V_k}$

$$\alpha_k \geq 0$$

THEOREM:

IF

$$0 \leq \alpha_k \leq \frac{1}{\psi_k^T P_{k-1} \psi_k} \quad (*)$$

THEN

* P STAYS POSITIVE DEFINITE

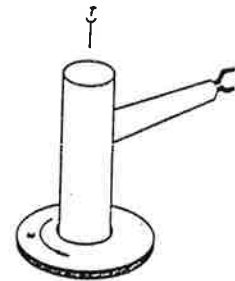
* $\bar{\theta}_k^T P_k^{-1} \bar{\theta}_k$ IS DECREASING \square

IN EACH ITERATION, TRY TO OBTAIN

$$\frac{x^T P_k x}{x^T x} = a \quad x = P_{k-1} \psi_k$$

BUT FULFIL THE LIMITATION (*)

EXAMPLE - INDUSTRIAL ROBOT



$$J \frac{d\omega}{dt} = T_e + T_f + T_b$$

$$\omega(k+1) = \omega(k) + \frac{1}{J} [T_e + T_f + T_b]$$

$$T_f = -k_f \text{sign}(\omega)$$

$$\theta = \left[\frac{1}{J} \quad \frac{k_f}{J} \right]^T$$

$$T_e = 0.5 \frac{1}{\hat{\theta}_1} (\omega_{ref} - \omega) + \frac{\hat{\theta}_2}{\hat{\theta}_1} \text{sign}(\omega)$$

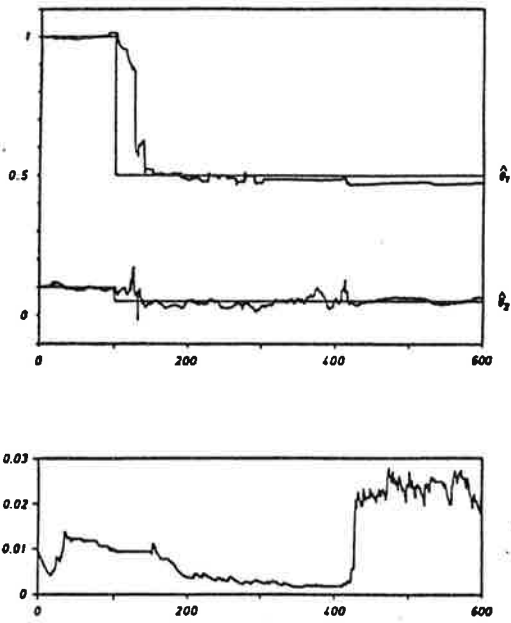


Figure 6.2 - The estimated parameters and the estimated noise variance.

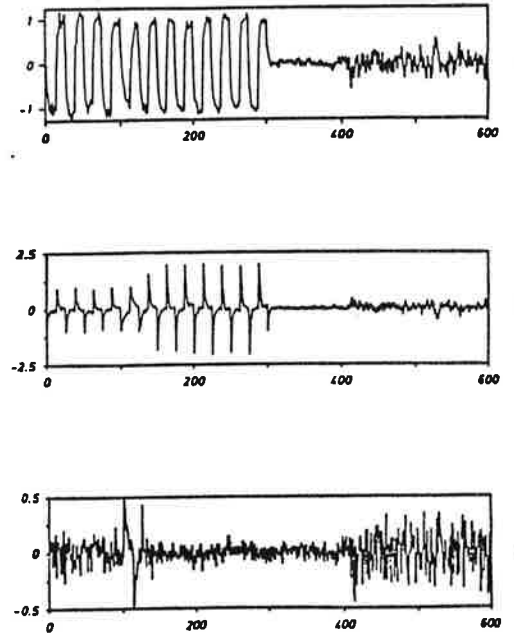


Figure 6.3 - The output- and input-signals of the system, and the residuals $e(t)$.

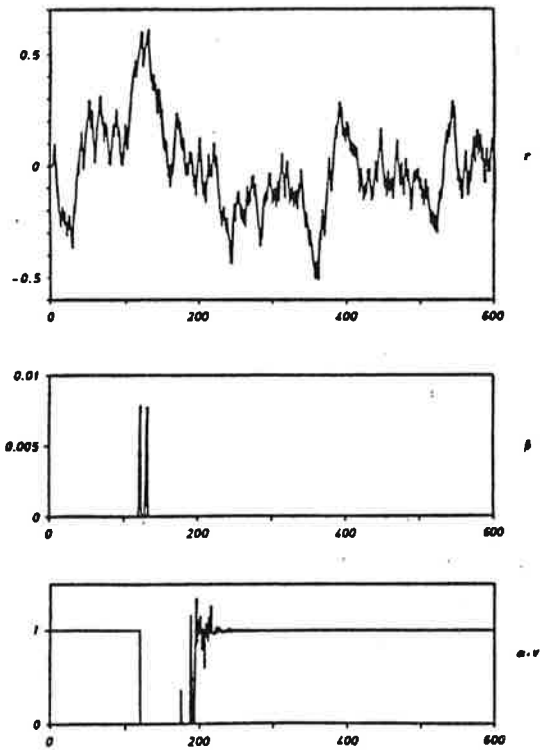


Figure 6.4 - The test signal $r(t)$, the additive gain $\beta(t)$ and the discounting measure $\alpha(t) \cdot v(t)$.

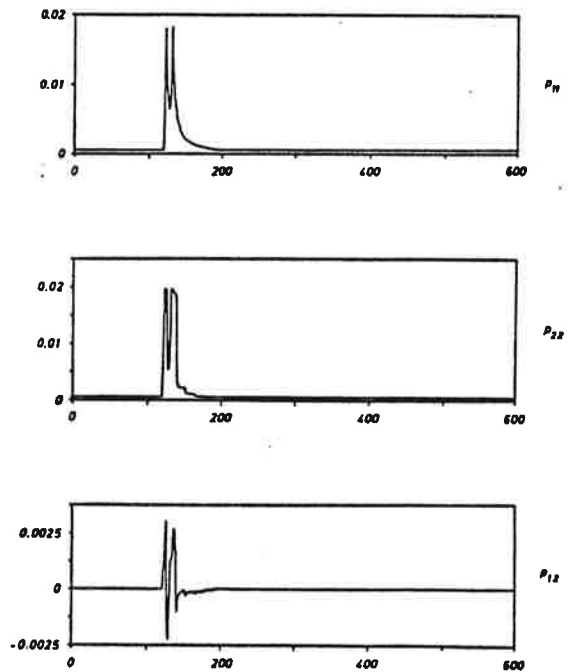
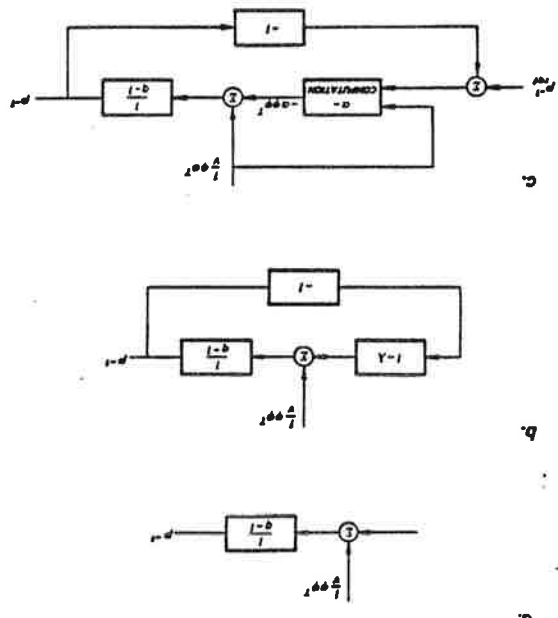


Figure 6.5 - The elements of the P-matrix.

FIGURE 1 - Block diagrams describing the updating of the inverse P-matrix.
a. The original LS procedure.
b. LS with forgetting factor.
c. The new proposed method.



7

7

7

Robustness of (MRAS) adaptive control

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The topic of this lecture was robustness of (MRAS) adaptive control subject to high frequency unmodeled dynamics of the process model. This area of stability problems can be classified in three categories: HCG-high controller gain, HAG-high adaptive gain and HF-high frequency input. Here only the adaptation loop is considered i.e. HAG- and HF-instabilities. In this way the resulting system to analys remains linear, time-varying.

The main idea is to approximate the high frequency dynamics with a right half plane zero. (Cf. Padé approximation of a timedelay.) Consider the example below. The approximation is valid for $\mu s \ll 1$.

$$G(s) = \frac{1}{s+1} \cdot \frac{1}{\mu s+1} = \frac{1}{s+1} \cdot \left(1 - \frac{\mu s}{1+\mu s}\right) \approx \frac{1}{s+1} \cdot (1-\mu s) = \frac{1+\mu}{s+1} - \mu \quad (1)$$

For stability analysis the differentiating effect of the RHP zero is stressed while for synthesis the throughput effect gives a clue how to improve robustness.

The MRAS scheme was analysed on the process above (1) using a first order reference model with unknown gain.

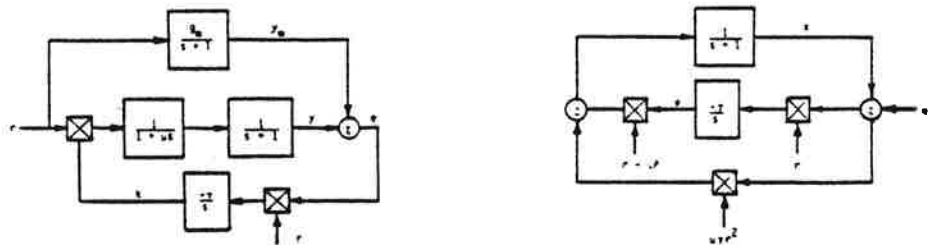


Fig. 1. The MRAS system and the transformed error system (2).

After the approximation (1) of the process and change of coordinates to to an error system, the following equations are obtained:

$$\frac{d}{dt} \begin{bmatrix} x \\ \psi \end{bmatrix} = \begin{bmatrix} -1 + \mu \gamma r^2 & r - \mu r \\ -\gamma r & 0 \end{bmatrix} \begin{bmatrix} x \\ \psi \end{bmatrix} + \begin{bmatrix} \mu \gamma r^2 \\ -\gamma r \end{bmatrix} e \quad (2)$$

where

x = difference in output between the actual and tuned system.
 ψ = difference in controller gain in reference to the tuned system.

HAG instability is shown in the following way. Assume $r=R$ const. and $e=0$. The characteristic equation gives the stability condition on the adaptive gain $\gamma R^2 < \mu^{-1}$.

To reveal HF-input instability, choose $r=R \sin(\omega t)$ and assume $\mu \gamma r^2 \ll 1$, $e=0$. Simulation shows a slow drift in ψ and a limit cycle in x . Approximation of the equations (2) for slow adaptation, i.e. $\gamma R^2 / \sqrt{1 + \omega^2}$ is sufficiently small, gives the stability condition $\omega^2 < \mu^{-1}$. One interpretation derived from the approximated equations is that instability is reached when the noise-to-signal ratio is larger than unity. Another interpretation is that the positive real condition for the plant is violated at high frequencies due to the RHP zero.

For the combined problem of HAG and HF stability, a detailed analysis shows that the joint stability condition is somewhat conservative.

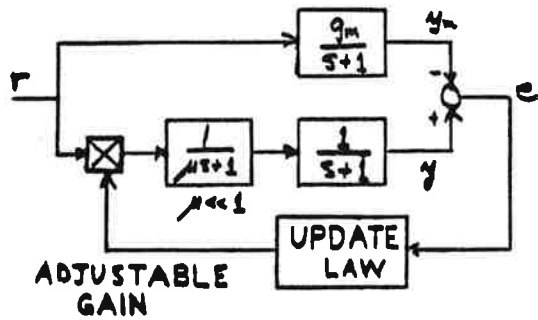
In the approximation (1) it is seen that the unmodeled high frequency dynamics can be seen as a negative throughput. A simple device to increase the stability properties of the MRAS scheme could therefore be to introduce a positive bypass β . Analysis by means of a Lyapunov function shows that stability is achieved for $\beta \geq \mu$. The positive real condition for the plant with bypass is also satisfied for these β .

The analysis shows that a bypass increases stability properties against unmodeled high frequency dynamics. However, a tracking error is introduced, which is difficult to predict when $\beta \neq \mu$. For instance, when β increases toward μ , the tracking error can decrease or increase depending on the frequency of the reference signal.

Reference

Kokotovic, P., Riedle, B.: Instabilities and stabilization of an adaptive system. Proc. American Control Conference, San Diego, California, June 6-8, 1984.

CAN THE SIMPLEST ADAPTIVE LOOP



BE UNSTABLE ?

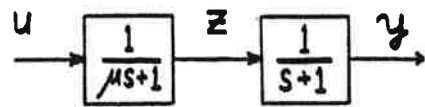
YES, IN TWO WAYS:

1. FAST (HAG)
2. SLOW (HF)

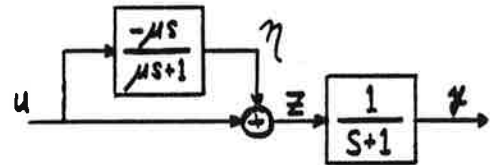
A METHOD TO STABILIZE THE LOOP

*(Discussions with
Aström
Bitmead
Rohrs
et al.)*

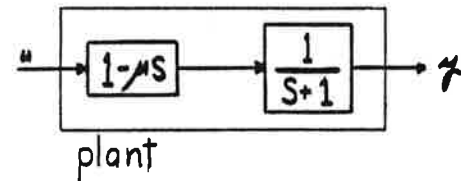
RIGHT-HALF-PLANE ZERO INSTEAD OF HF PARASITICS



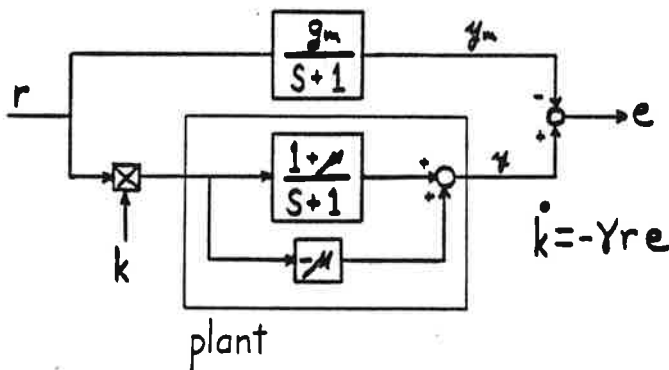
$$\begin{aligned} \dot{y} &= -y + z \\ \mu \dot{z} &= -z + u \end{aligned} \iff \begin{aligned} \dot{y} &= -y + u + \eta \\ \mu \dot{\eta} &= -\eta - \mu \dot{u} \end{aligned}$$



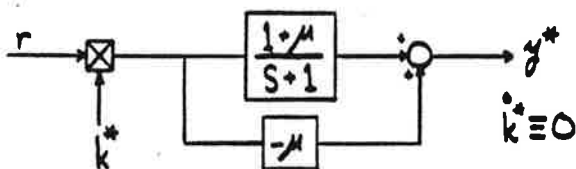
AT LOW FREQUENCIES, LET $\eta = -\mu \dot{u}$



ADAPTIVE SYSTEM



TUNED SYSTEM



TUNED ERROR

$$e^* \triangleq y^* - y_m$$

FACTS FROM STABILITY THEORY

FACT 1

$$\lim_{T \rightarrow \infty} \int_0^T \text{trace } A(t) dt = +\infty$$

IS SUFFICIENT FOR INSTABILITY OF

$$\dot{x} = A(t)x \quad \square$$

FACT 2

IF $\dot{x} = A(t)x$ IS EXPONENTIALLY STABLE, THEN

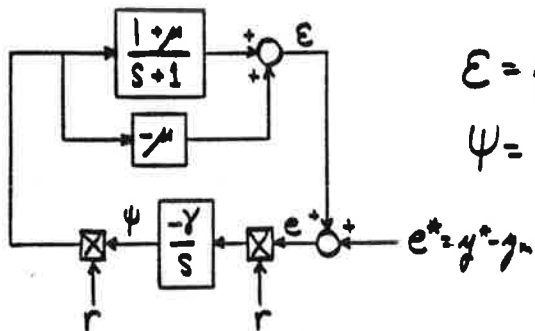
$$\dot{x} = (A(t) + B(t))x$$

IS EXPONENTIALLY STABLE WHEN

$$\int_0^t \|B(\tau)\| d\tau < c, t + c_0, \forall t > c_0$$

FOR SOME CONSTANTS c , and c_0 !

FAST ADAPTATION ($\mu \dot{r}$ SMALL)



$$\epsilon = y - y^*$$

$$\psi = k - k^*$$

LINEAR, LET $e^* = 0$

$$\begin{bmatrix} \dot{\epsilon} \\ \dot{\psi} \end{bmatrix} = \begin{bmatrix} -1 + \mu \gamma r^2 & r \\ -\gamma r & 0 \end{bmatrix} + \begin{bmatrix} 0 & -\mu \dot{r} \\ 0 & 0 \end{bmatrix} \begin{bmatrix} \epsilon \\ \psi \end{bmatrix}$$

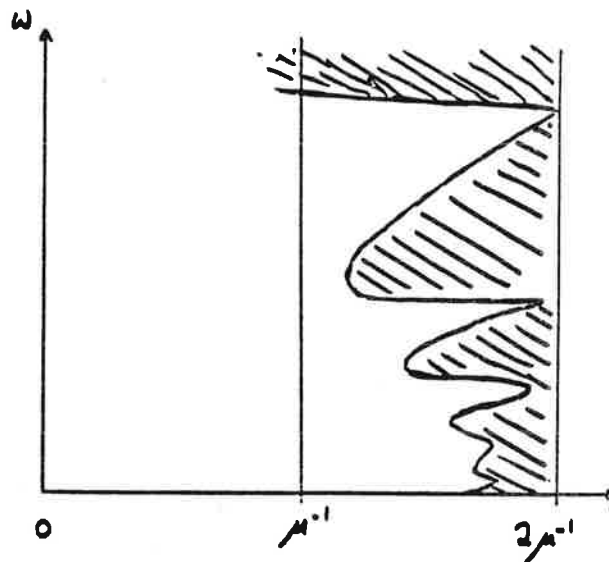
NECESSARY FOR STABILITY

$$\mu \gamma \lambda_1 < 1, \quad \lambda_1 \triangleq \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T r^2(\tau) d\tau \quad (= \text{average of } r^2(\tau))$$

SUFFICIENT FOR STABILITY ($\mu \dot{r}$ SMALL)

$$\mu \gamma r^2(t) < 1 \quad \forall t$$

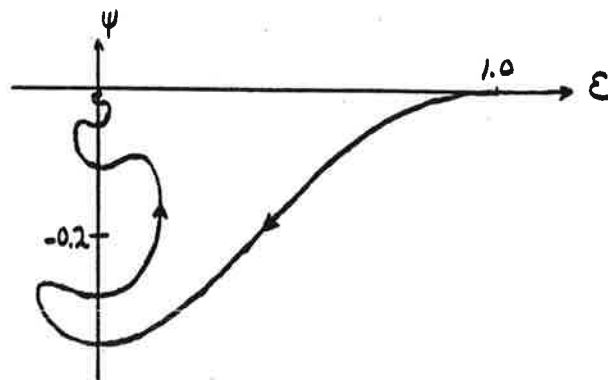
$$V_1 = \frac{1}{2} \epsilon^2 + \frac{1}{2\gamma} \psi^2, \quad \dot{V}_1 = (-1 + \mu \gamma r^2(t)) \epsilon^2$$



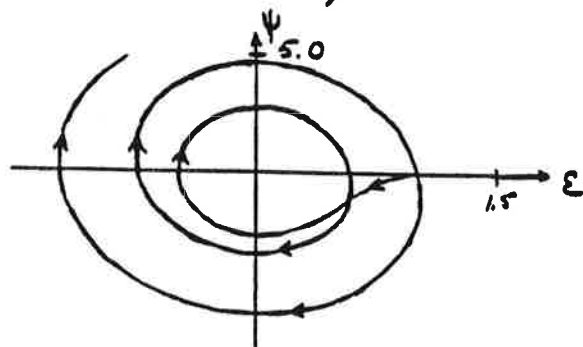
$$r = \sin \omega t$$

$\mu \gamma < 1$
sufficient
($\mu \omega$ small)

$\frac{\mu \gamma}{2} < 1$
necessa

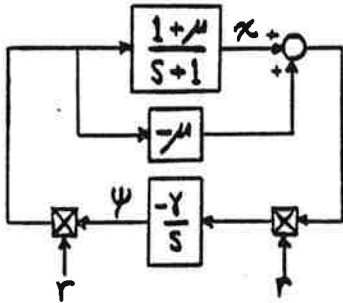


$$r = \sin t \quad \mu = 0.1 \quad \gamma = 1$$



$$r = \sin .5t \quad \mu = 0.1 \quad \gamma = 24$$

SLOW ADAPTATION (γ and $\mu \gamma r^2$ SMALL)



$$\dot{\chi} = -\chi + \psi(1+\mu)r$$

$$\dot{\psi} = \gamma(-r\chi + \mu r^2)$$

$$r(t) \rightarrow \frac{1+\mu}{s+1} \rightarrow c(t) \quad c(t) \triangleq \int_0^t (1+\mu) e^{-(t-\tau)} r(\tau) d\tau$$

WHEN $\gamma=0$

$$\chi(t) = c(t) \psi(t), \quad \psi(t) = \text{const.}$$

DEFINE $\delta(t)$ SUCH THAT

$$\chi(t) = c(t) \psi(t) + \gamma \delta(t)$$

$$\mu < \frac{\lambda_2}{\lambda_1}$$

REQUIRES LOW FREQUENCY REFERENCE INPUTS

$$r(t) = \sum_{i=1}^N R_i \sin(\omega_i t + \theta_i)$$

$$\mu < \frac{\lambda_2}{\lambda_1} = \frac{\sum_{i=1}^N \frac{1+\mu}{1+\omega_i^2} R_i^2}{\sum_{i=1}^N R_i^2}$$

$$\sum_{i=1}^N \frac{1}{1+\frac{1}{\mu}} R_i^2 < \sum_{i=1}^N \frac{1}{1+\omega_i^2} R_i^2$$

Sufficient $\frac{1}{\mu} > \omega_i^2, \quad \omega_i < \frac{1}{\sqrt{\mu}}$
 $\forall i \in [1, N]$

SLOW ADAPTATION (γ and $\mu \gamma r^2$ SMALL)

$$\begin{bmatrix} \dot{\delta} \\ \dot{\psi} \end{bmatrix} = \begin{bmatrix} -1+\gamma r c & (c-\mu r) r c \\ 0 & \gamma(\mu r^2 - r c) \end{bmatrix} + \begin{bmatrix} 0 & 0 \\ -\gamma r & 0 \end{bmatrix} \begin{bmatrix} \delta \\ \psi \end{bmatrix}$$

$$\dot{\delta} = (-1+\gamma r(t) c(t)) \delta \quad \dot{\psi} = \gamma(\mu r^2(t) - r(t) c(t)) \psi$$

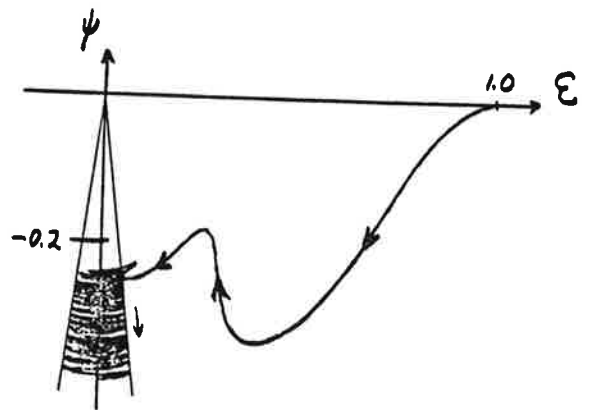
STAB. COND. $-1+\gamma \lambda_2 < 0$ $\gamma(\mu \lambda_1 - \lambda_2) < 0$

$$\lambda_1 = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T r^2(t) dt \quad \lambda_2 = \lim_{T \rightarrow \infty} \frac{1}{T} \int_0^T r(t) c(t) dt$$

= (average of r^2) = (average of $r c$)

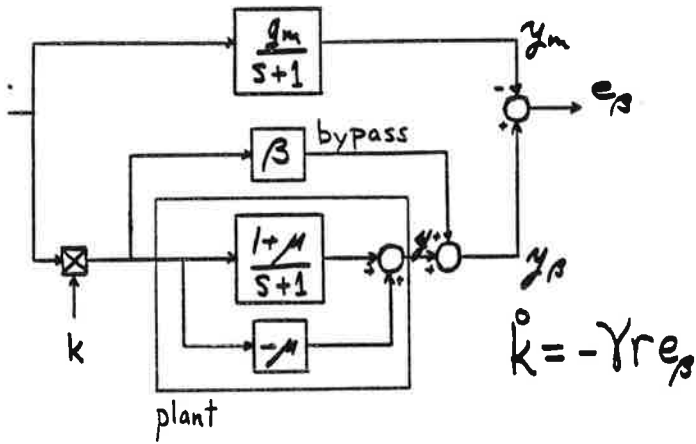
NECESSARY AND SUFFICIENT CONDITION FOR SLOW ADAPTATION STABILITY

$$\mu < \frac{\lambda_2}{\lambda_1} = \frac{\text{average of } r c}{\text{average of } r^2}$$

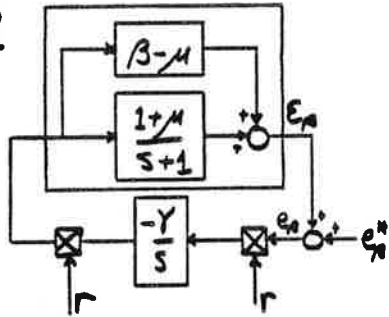


$$r = \sin 4t \quad \mu = 0.1 \quad \gamma = 1.0$$

STABILIZATION WITH BYPASS



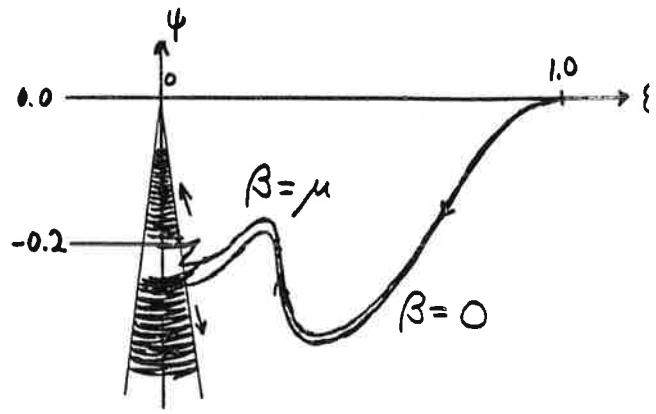
SPR
for
 $\beta \geq \mu$



$$e_B = y_B - y_B^*$$

$$\psi = k - k^*$$

$$e_B^* = y_B^* - y_B$$



$$r = \sin 4t \quad \gamma = 1.0 \quad \mu = 0.1$$

CONCLUDING REMARKS

* SIMPLEST ADAPTIVE SYSTEM EXHIBITS TWO TYPES OF INSTABILITY CAUSED BY ADAPTATION

* THESE TWO MECHANISMS PLUS LINEAR HIGH CONTROLLER GAIN CAN EXPLAIN ALL PHENOMENA OBSERVED IN MORE COMPLEX ADAPTIVE SYSTEMS

* BOUNDS DERIVED HERE WILL GENERALIZE

$$\gamma r^2 < 1 \rightarrow \mu \omega^T \Gamma \omega < 1$$

$$\text{ave}(rc - \mu r^2) > 0 \rightarrow \text{ave}(\omega c^T - \mu \omega \omega^T) > 0$$

Parameter convergence issues in MRAC

S. Shankar Sastry
Dept. of Elect. Eng. and Comp. Sciences
University of California
Berkeley, USA

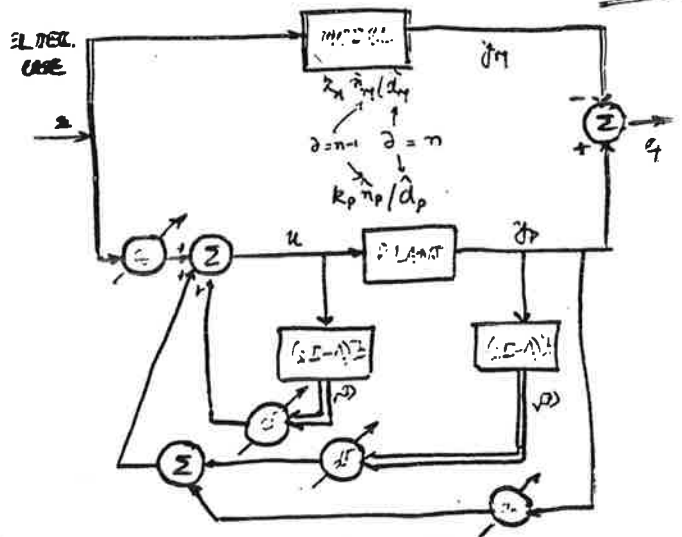
This contribution addresses parameter convergence in continuous time model reference adaptive control. The scheme of Narendra et al. is considered. The reported result can be summarized as follows:

- The technique of generalised harmonic analysis is employed to translate the persistent excitation condition on the signal vector into an equivalent condition on the exogenous reference signal only for parameter convergence.
- Partial convergence results for the case where the reference signal is not sufficiently rich are derived.
- Connections with robustness are illustrated by a first order example with output plant disturbance and plant parameter variation.

PARAMETER CONVERGENCE ISSUES IN MODEL REFERENCE ADAPTIVE CONTROL

S. S. SASTRY *Joint work with* S. Boyd
 M. Bodson
 UNIVERSITY OF CALIFORNIA
 BERKELEY

NECESSARY CONDITIONS FOR PARAMETER CONVERGENCE



PARAMETER VECTOR $\theta^T = [c_0, c^T, d_0, d^T] \in \mathbb{R}^{2m}$
 SIGNAL VECTOR $w = [z, v^{(1)}, y, v^{(2)}] \in \mathbb{R}^{2m}$
 CONTROL LAW $u = \theta^T w$ UPDATE $\dot{\theta} = -\epsilon w$

-1st block $\Rightarrow y \rightarrow 0$ as $t \rightarrow \infty$
 This is not true.
 2nd block.

NOTHING CAN BE SAID ABOUT CONVERGENCE OF θ .

$\exists \theta^* \in \mathbb{R}^{2m}$ such that $\theta^* = [c_0^*, c^{*T}, d_0^*, d^{*T}] \in \mathbb{R}^{2m}$
 L-Plant transf. $f_m =$ model transfer function
 $\phi = G - G^*$ PARAMETER ERROR

PERMAN (ANDERSON, MOHAMMADREZA, ...)

$\theta, e \rightarrow 0$ exponentially $\Leftrightarrow \exists \alpha, \delta > 0 \Rightarrow$

$$\int_s^{s+\delta} w w^T dt \geq \alpha I \quad \forall s. \quad \text{PERSISTENT EXCITATION}^2$$

$$w^T = [z, v^{(1)}, y, v^{(2)}]$$

REMARKS: (1) Condition not explicit since it is on signals inside time-varying adaptive loop.

RESTRICTIONS: (1) CONDITIONS ON EXOGENOUS REF SIGNAL

- (1) for parameter convergence
- (2) PARTIAL CONVERGENCE RESULTS
- (3) CONNECTIONS WITH ROBUSTNESS

TECHNIQUE: GENERALIZED HARMONIC ANALYSIS.
 A LA WIENER

Generalized Harmonic Analysis
 $w: \mathbb{R}_+ \rightarrow \mathbb{R}^m$ is said to have autocovariance

$$R_w(\tau) \in \mathbb{R}^{m \times m} \quad \lim_{T \rightarrow \infty} \frac{1}{T} \int_s^{s+T} w(t) w^T(t+\tau) dt = R_w(\tau)$$

uniformly in s (1).

def: w is persistently exciting $\Leftrightarrow R_w(0) > 0$.

def: $R_w(\tau)$ has the Bochner (spectral) representation

$$R_w(\tau) = \int e^{i\omega\tau} S_w(d\omega) \quad \text{SPECTRAL MEASURE} \square$$

$$\bullet \text{ (a) } R_w(\tau) > 0 \Leftrightarrow \int S_w(d\omega) > 0$$

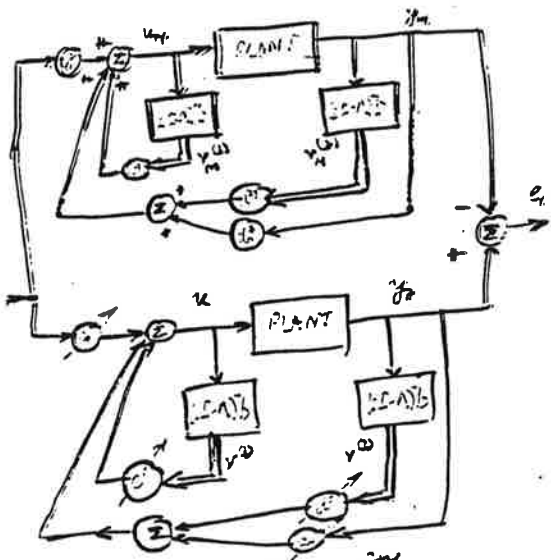
LEMMA 3 LINEAR FILTER LEMMA

Let $w: \mathbb{R}_+ \rightarrow \mathbb{R}^m$ have spectral measure $S_w(d\omega)$ and h an $m \times m$ matrix of bounded measure. Then $f = h * w$ has spectral measure

$$S_f(d\omega) = h(\omega) S_w(d\omega) h^*(\omega)$$

LEMMA 4 TRANSIENT LEMMA

$$u - v \in L^2 \Rightarrow R_u(\tau) = R_v(\tau)$$



$$v^T = [v_1, v_2, \dots, v_n] \in \mathbb{R}^n$$

$$w^T = [w_1, w_2, \dots, w_n] \in \mathbb{R}^n$$

idea $\Rightarrow w \in \mathbb{C} \Rightarrow R_{ww}(s) = R_{ww}(s)$

$$R_{ww}(s) > 0 \Leftrightarrow R_{ww}(s) > 0 \Leftrightarrow \int S_{ww}(d\nu) > 0$$

matrix function associated with \$S\$ which is

$$\begin{bmatrix} I & \hat{A} \\ \hat{A}^T & I \end{bmatrix} \begin{bmatrix} \hat{A} \\ I \end{bmatrix}^{-1} b = \hat{Q}^T(s)$$

$$S_{ww}(d\nu) = \hat{Q}(j\nu) \hat{Q}^*(j\nu) S_n(d\nu)$$

(Supp \$S_n\$) \$\ge\$ \$2n\$ points.

$$\hat{a}(s) \hat{d}_p(s) + \hat{b}(s) \hat{n}_p(s) \equiv 0$$

$$\frac{\hat{n}_p}{\hat{d}_p} = \frac{-\hat{a}^{-n-2}}{\hat{b}^{-n}}$$

Coprimeness of \$\hat{n}_p, \hat{d}_p\$

note: 1. we have shown that when reference signal is sufficiently rich Supp \$S_n(d\nu) \ge\$ \$2n\$ points then we have parameter convergence.

2. Only assumption: \$r\$ have autocorrelation

Rate of convergence: let \$a', s'\$ be such that

$$\int_s^{s+s'} w \cdot (\hat{A}w)^T dt \geq a' I \quad \forall s \in \mathbb{R}_+$$

connection with location of spectrum \$S_n(d\nu)\$??

then, Lyapunov exponent of \$\phi\$ parameter vector

$$\log \frac{1+a'}{2s'} \approx \frac{a'}{2s'} \text{ for small } a'$$

Lemma: \$w_{t+1}\$ is persistently exciting \$\Leftrightarrow\$ Spectral measure of \$r_t\$ is rate concentrated on \$k < 2n\$ points.

\$\Rightarrow\$ Assume that \$S_n\$ is conc. on \$\nu_1, \dots, \nu_k\$ where \$k < 2n\$

Then:

$$R_{ww}(0) = \int \hat{Q}(j\nu) S_n(d\nu) \hat{Q}^*(j\nu) = \sum_{m=1}^k \frac{\hat{Q}(j\nu_m) \hat{Q}^*(j\nu_m)}{S_n(j\nu_m)}$$

Since it is the sum of \$k < 2n\$ dyads,

\$R_{ww}(s)\$ is singular. Assume that \$w\$ is not p.e. then, \$\exists c \in \mathbb{R}^n \neq 0\$

$$0 = c^T R_{ww}(s) c = \int |\hat{Q}(j\nu)^T c|^2 S_n(d\nu)$$

i.e. \$\hat{Q}^T(j\nu) c = 0 \quad \forall \nu \in \text{Supp}(S_n)\$

Now \$\exists\$ constant matrix \$M \Rightarrow\$

$$\hat{Q}^T(s) M = \frac{1}{s^p} \begin{bmatrix} \hat{d}_p & \dots & \hat{d}_p^{n-2} & \hat{n}_p & \dots & \hat{n}_p^{n-1} \end{bmatrix}$$

then we have

$$\hat{Q}^T(j\nu) M \tilde{c} = 0 \quad \forall \nu \in \text{Supp}(S_n)$$

i.e. \$\exists\$ poly. \$\hat{a}(s)\$ & \$\hat{b}(s)\$ of \$\ge\$ \$n-2\$, and \$n\$ resp \$\in\$ \$\mathbb{C}^n\$ s.t. \$\hat{a}(s) \hat{d}_p(s) + \hat{b}(s) \hat{n}_p(s) = 0 \quad \forall \nu \in \text{Supp}(S_n)\$

PARTIAL CONVERGENCE

If \$S_n\$ is concentrated on \$k < 2n\$ points \$\nu_1, \dots, \nu_k\$. Then let \$\Theta\$ be the set of \$\Theta_i\$ for which e.c. plant matches model i.e.

$$\begin{bmatrix} \hat{Q}(j\nu_1) \hat{Q}(j\nu_1)^T \\ \vdots \\ \hat{Q}(j\nu_k) \hat{Q}(j\nu_k)^T \end{bmatrix} \Theta = \begin{bmatrix} \hat{n}(j\nu_1) \\ \vdots \\ \hat{n}(j\nu_k) \end{bmatrix}$$

$$\Theta = \Theta^* + \text{Nullspace} \begin{bmatrix} \hat{Q}(j\nu_1) \hat{Q}(j\nu_1)^T \\ \vdots \\ \hat{Q}(j\nu_k) \hat{Q}(j\nu_k)^T \end{bmatrix}$$

$$= \Theta^* + \text{NullSpace } R_{ww}(s)$$

\$\Theta \rightarrow \Theta\$ as \$k \rightarrow \infty\$, \$\Theta\$ itself need not converge but \$\text{dist}(\Theta, \Theta^*) = 0\$ as \$k \rightarrow \infty\$

Rate of convergence may not be exponential.

BLK & RED. DEGREE CASES

Red. degree 2 same statement - parameter convergence iff $\text{Supp } S_n(\lambda) \geq 2n$ points.

Red. degree ≥ 3 - new parameter ϵ_{2n+1} associated with gain parameters for augmented error signal - being partial convergence theorem, ϵ_{2n+1} all but ϵ_{2n+1} converge iff $\text{Supp } S_n(\lambda) \geq 2n$ points. (!!!). WATCH OUT FOR ϵ_{2n+1} in the augmented error scheme.

CONNECTIONS WITH ROBUSTNESS

W. (VIN. MASARU - 1/1/1982)

Consider $\dot{x} = f(x, u, t)$, locally Lipschitz in x, u .

Then, if

$\dot{x} = f(x, u, t)$ is exponentially stable (uniformly)

$\Rightarrow \dot{x} = f(x, u, t)$ is small signal BIBO stable with $x(0) = 0$.

is $\exists c \exists \epsilon \|u\|_{\infty} \leq \epsilon \Rightarrow \|y\|_{\infty} \leq k \cdot \epsilon$

REVISION: If $x(0) \neq 0$, $\forall \epsilon > 0 \exists T_{\epsilon} > 0$

$\|y\|_{\infty} \leq k(1+\epsilon) \|x\|_{\infty}$ for $t \geq T_{\epsilon}$.

REMARKS: 1) EXPONENTIAL STABILITY IS ABSOLUTELY NECESSARY FOR THIS THEOREM TO WORK - NUMEROUS COUNTEREXAMPLES EXIST.

How about red degree ≥ 3 , with augmented error?

2) L_{∞} gain k depends on rate of exponential convergence (inversely); and so does T_{ϵ} .

Application

1ST ORDER PLANT MODEL
 $\dot{y}_p = -a_p y_p + b u$ (UNKNOWN)
 $\dot{y}_m = -a_m y_m + b u$
 CONTROL $u = z + d y_p$
 $\dot{d} = -e_1 \frac{y_p}{s}$
 $e_1 = y_p - 1 u$
 $\phi = 1 - d^2$
 PARAMETER ERROR

$$\begin{pmatrix} \dot{e}_1 \\ \dot{y} \end{pmatrix} = \begin{pmatrix} -a_m & y_p(s) \\ -y_p(s) & 0 \end{pmatrix} \begin{pmatrix} e_1 \\ d \end{pmatrix} = \begin{pmatrix} -a_m e_1 + \phi y_m + \phi e_1 \\ -e_1 - e_1 y_m \end{pmatrix}$$

$\dot{x} = f(x, t)$ EXP. CONVER. IFF $\text{Supp } S_n \geq 1$ pts

%p DISTURBANCES

$$\begin{aligned} \dot{e}_1 &= -a_m e_1 + \phi y_m + \phi e_1 + \phi p + d^2 p \\ \dot{d} &= -e_1 - e_1 y_m - \frac{2a_m e_1 - b y_m - p^2}{s} \end{aligned}$$

1ST ORDER PARAMETER VARIATION

$$\begin{aligned} \dot{e}_1 &= -a_m e_1 + \phi y_m - \phi e_1 \\ \dot{d} &= -e_1 - e_1 y_m - a_0 \end{aligned}$$

$$\dot{x} = f(x, t) + g(x, t) \begin{pmatrix} 0 \\ a_p \end{pmatrix}$$

UNMODELLED DYNAMICS - CHOICE OF MODEL CRUCIAL. TECHNIQUE - CONVERT TO %p DISTURBANCES. NOT THAT CAN BE HANDLED DEPENDS ON EXTENT OF ...

On adaptive control with prescribed robustness properties

Eva Trulsson

Department of Electrical Engineering
University of Linköping
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We will describe a frequency domain method for regulator design. The resulting regulator has a stability margin in terms of the Nyquist curve which is independent of the system. This is a useful property in an adaptive control application, where the controller at every step is designed for a recursively identified model of the system, for the following reason. Since the Nyquist curve is known it is known at which frequencies it is important to have a good model of the system in order to have a good control result. Then the input-output data can be filtered through filters which emphasizes these frequencies and this will lead to a model of the system which is best there. (See B. Wahlberg and L. Ljung (1984)).

The main idea of the method is to obtain a given Nyquist curve by a cancellation of the system poles and zeroes. Therefore the basic version can only be applied to stable minimum phase systems. By a slight modification it is however possible to handle also pure integrators and discrete time real unstable zeroes.

Reference

B. Wahlberg and L. Ljung (1984): Design variables for bias distribution in transfer function estimation. Internal report, Department of Electrical Engineering, Linköping University, S-581 83 Linköping, Sweden.

On an adaptive control algorithm with prescribed robustness properties.

- There will always be unmodeled dynamics.
- Most important to have a good model of the system around the cross-over frequency.
- Filtering can be used to distribute the bias

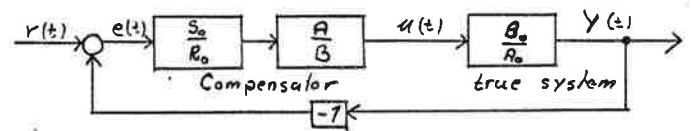
A frequency domain method.

Poleplacement: The Nyquist curve depends almost as much on the system as on the desired closed loop poles.

That may be a problem in an adaptive control application.

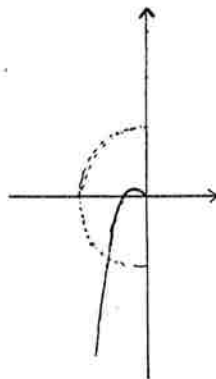
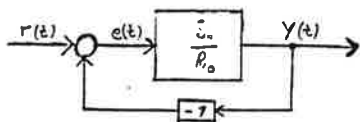
Use the control:

$$u(t) = -\frac{S_0}{R_0} \cdot \frac{A}{B} (Y(t) - r(t)) \quad (*)$$



Another design method.

Choose some 'ideal' open loop reference system S_0/R_0 .



Observations

- Some robustness against multiplicative errors in $\frac{S_0}{R_0}$ for all systems. (Given by $\frac{S_0}{R_0}$)
- In order to have (*) realizable $\frac{S_0}{R_0}$ and $\frac{A}{B}$ must be of the same type. (Same pole-excess or same number of delays.)
- Only stable minimum phase systems can be handled. (With some modifications also integrators and real discrete time zeroes can be treated.)

on minimum phase zeroes.

Suppose that

$$\frac{B}{A} = \frac{z^{-d} B_I B_S}{A}$$

where

$$B_I = k \cdot (1 + bz^{-1}) \quad b > 0$$

Then choose S_0 with $d+1$ pure delays.

$$\frac{S_0}{R_0} = \frac{z^{-d-1} S_1}{B_S}$$

as reference loop gain.

and use the control

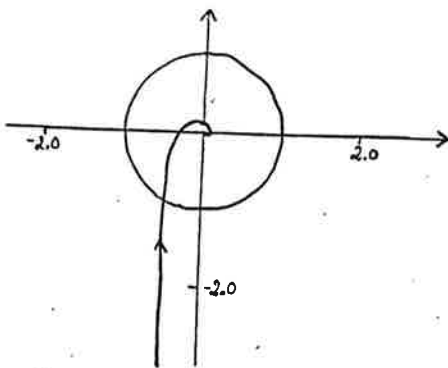
$$u(t) = -\frac{S_1 A}{R_0 B_S} (y(t) - r(t))$$

This will lead to the loop gain

$$G_u \cdot G = \frac{S_1 A}{R_0 B_S} \cdot \frac{B_I B_S z^{-d}}{A} = \underbrace{\frac{S_1 z^{-d-1}}{R_0}}_{\text{desired gain}} \cdot \underbrace{z B_I}_{\text{nice}}$$

Reference open loop system:

$$G(z^{-1}) = \frac{0.125 z^{-2} (1+z^{-1})}{(1-z^{-1})}$$



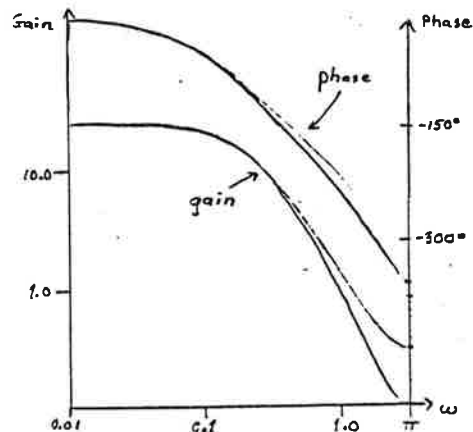
- $\pm 0^\circ$ phase margin
- integral action
- gain decreasing to zero
- poles at -0.27 ; $0.65 = 0.25z$ "p"

An example.

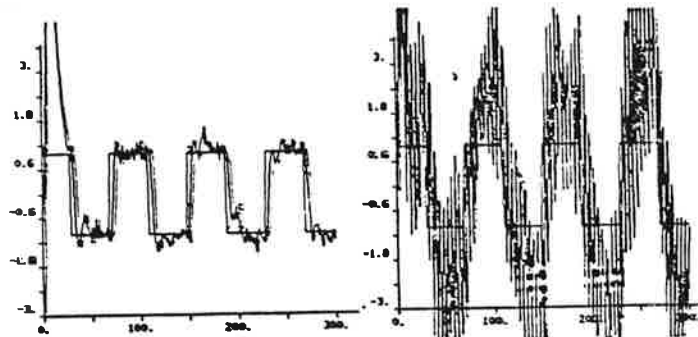
System:

$$Y(z) = \frac{z^{-2} (1+0.25z^{-1})}{(1-0.8z^{-1})^2 (1-0.25z^{-1})} u(z) + e(z)$$

Almost second order

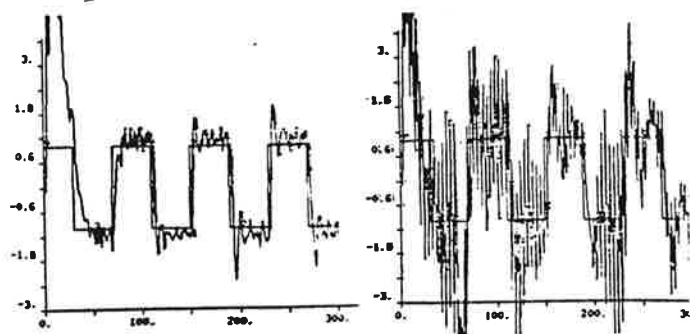


— true system
— a second order system



Reference system

poleplacement in the origin



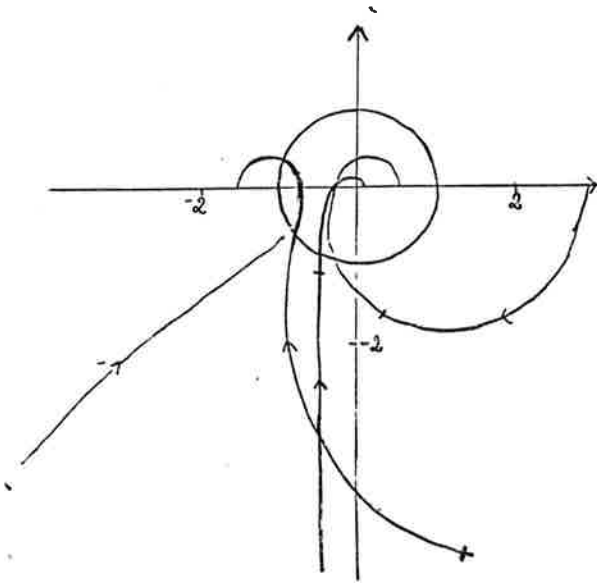
poleplacement at "p"

poleplacement at "p" with integral action

$$p = -0.27 ; 0.65 = 0.25z$$

Conclusions

- Frequency domain (robustness) properties which are independent of the system model.
- No problems with common factors in A and B .
- Simple.
- The disturbance rejection properties may be bad.
- The choice of reference system may need some initial testing.



- pole placement in the origin
- pole placement at " p "
- reference system
- pole placement at " p " with integral action.

$$p = -0,27 ; 0,63 \pm 0,25j$$

On living with the positive real condition

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In this lecture an example is given on a simple model reference adaptive system for which instability is achieved for certain command signals. A modification of the adaptation is suggested to overcome the problem.

In the example the reference model is of 1st order while the process is of 3rd order. The adjustable parameters are the feedforward gain k_v and the feedback gain k_f . If $k_v = k_v^*$ is known and the reference signal $r(t)$ is a step r_0 then the closed loop system is linear and time invariant. It is then easy to see that for r_0 sufficiently large the closed loop system will become unstable unless the process transfer function is positive real. An easy way to eliminate this problem is to use a normed adaptation gain $g_r = g_v / r^2$ instead of g_v . If the reference signal is a sinusoidal $r(t) = \sin \omega t$ then simulations show that for frequencies larger than a certain frequency ω_0 the system will be driven into instability.

The proposed method is based on frequency domain arguments. The idea is to turn off the adaptation for sufficiently high frequencies in the reference input. For these frequencies the adaptation is done with respect to a benign model with positive real transfer function $G_{\text{good}}(s)$.

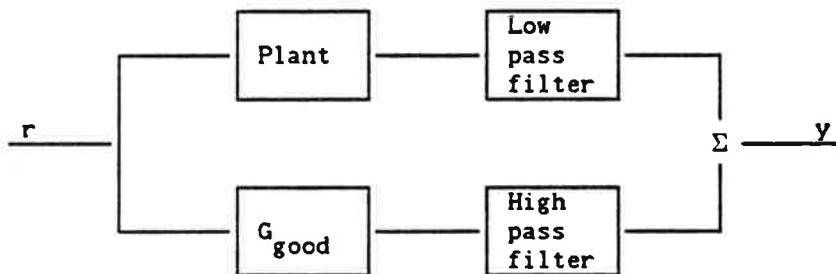


Fig. The "conditioned plant".

The following assumptions are made :

- The sign of the process gain is known.
- The process is minimum phase.
- The relative degree of the process is known.
- The controller uses $2n^*$ parameters, where n^* is an upper bound on the process order.

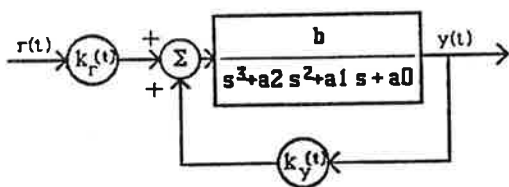
Simulations of the MRAS when applied to the "conditioned plant" show that the parameter drifts due to high frequency command signals are eliminated. There are however no theoretical results on the robustness of the modified controller.

Living with Positive Realness

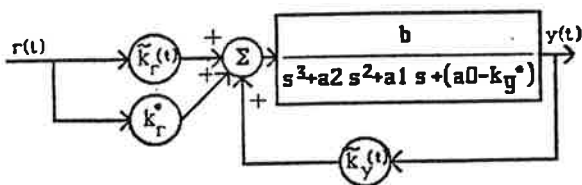
Charles E. Rohrs
Univ. of Notre Dame

1. The positive real condition appears to be necessary as well as sufficient
2. There are measures which can be taken which may be taken to allow us to live with the positive real condition.

1. Excitation
2. Plant Conditioning



The adaptive controllers setup



$$P^* = \frac{b}{s^3 + a_2 s^2 + a_1 s + (a_0 - k_g^*)}$$

Adaptive controller's error system

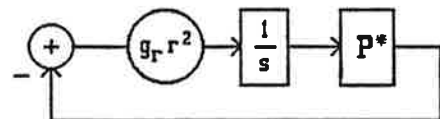
Case 1

k_g^* known; $r(t) = r$, a constant

$$\frac{d}{dt} \begin{bmatrix} e \\ \dot{e} \\ \ddot{e} \\ \tilde{k}_r \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ -(a_0 - k_g^*) & -a_1 & -a_2 & br \\ -g_r r & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} e \\ \dot{e} \\ \ddot{e} \\ \tilde{k}_r \end{bmatrix}$$

Stability determined by characteristic equation

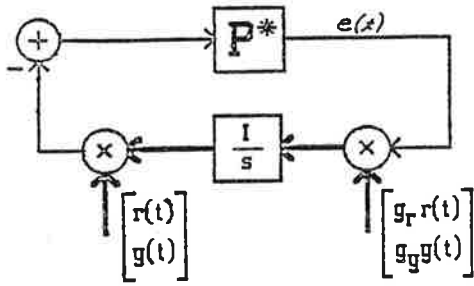
$$s(s^3 + a_2 s^2 + a_1 s + (a_0 - k_g^*)) + b g_r r = 0$$



$\exists r$ which will cause instability iff $\arg P^*(j\omega) > 90$ degs for some ω

One should use $g_r = \frac{g_r^*}{r^2(t)}$

ERROR SYSTEM CAN BE VIEWED AS FOLLOWS:



If true plant were first order, then there would be feedback parameter k_y^* so that P^* would be positive real or passive.

Then the passivity theorem would say that the loop is stable.

For the case where the plant is properly modeled, the stability result holds despite the fact that the feedback portion of the loop may have large gain.

Example: Assume $r(t)$ and $e(t)$ are sinusoid of the same frequency.

CASE 2

k_y^* known, $r(t)$ sinusoidal
Linear, periodic $A(t)$

$$\frac{d}{dt} \begin{bmatrix} e \\ \dot{e} \\ \ddot{e} \\ \tilde{k}_T \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ -(a_0 - k_y^*) & -a_1 & -a_2 & b r(t) \\ -g_T r(t) & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} e \\ \dot{e} \\ \ddot{e} \\ \tilde{k}_T \end{bmatrix}$$

$$x(t) = P(t)e^{Kt} x(0)$$

$$P(t) = P(t+T)$$

$$a_0 - k_y^* = 2; a_1 = 5; a_2 = 4; b = V_r = 1$$

$$r(t) = \sin \omega_0 t$$

$$\omega_0 = 2.3 \quad \arg P^*(2.3) = 180^\circ \quad \lambda_{\max}(e^{k^*t}) = 1.08$$

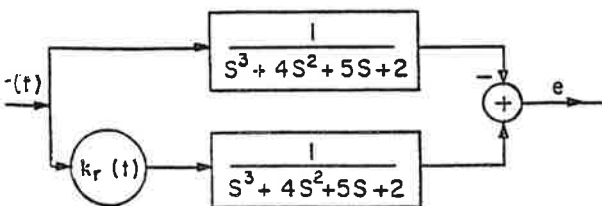
$$\omega_0 = .71 \quad \arg P^*(.71) = 90^\circ \quad \lambda_{\max}(e^{k^*t}) = 1.007$$

$$\omega_0 = .70 \quad \arg P^*(.70) = 90^\circ \quad \lambda_{\max}(e^{k^*t}) = .984$$

Note that this instability is independent of the reference model used to form the error.

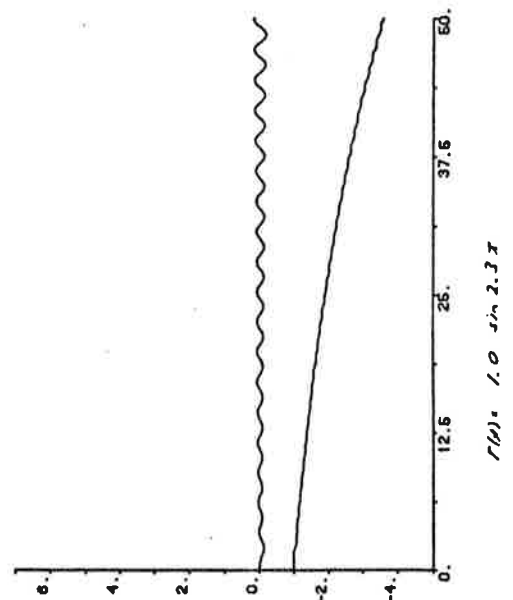
The instability cannot be explained by any inability of the adaptive system to match the model with a well behaved nominal system.

For example, Case 2 could have arisen from the following system.



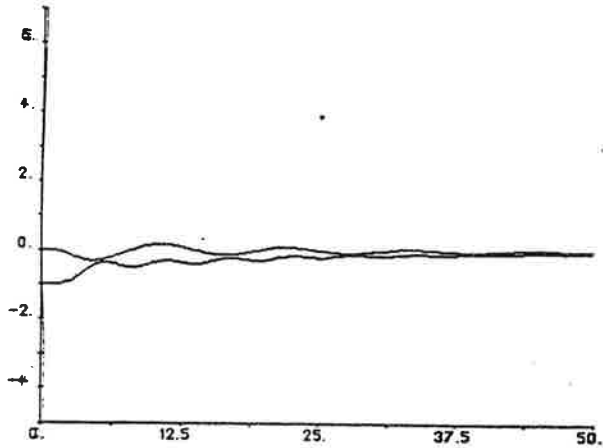
We know from Parks that creating a positive real operator is sufficient for stability. These results indicate that the positive real condition is necessary.

84.08.14 - 08:44:38 p. 1
hoopy



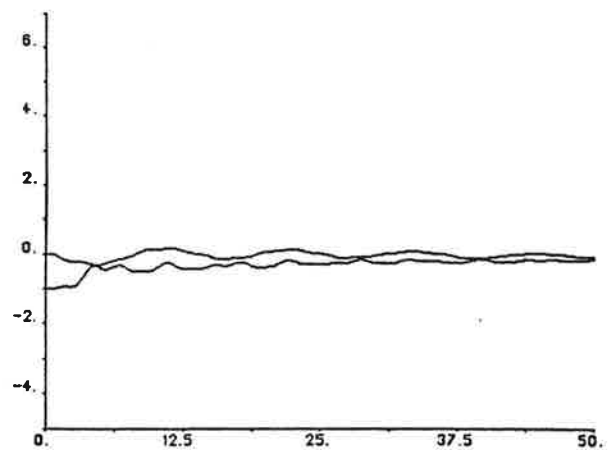
$r(t) = 1.0 \sin 2.3 t$

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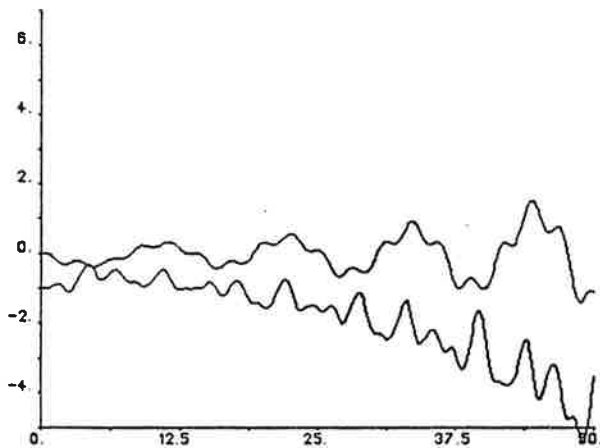
$$r = 1.0 \sin .555t$$

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$$r = 1.0 \sin .555t + 1.0 \sin 2.3t$$

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$$r = 1.0 \sin .555t + 1.0 \sin 2.3t$$

Assumptions needed in stability proofs of adaptive algorithms:

1. The sign of g_p is known
2. The zeroes of $B(s)$ are in the left half plane
3. The relative degree of the plant is known exactly
4. The controller uses $2n^*$ parameters where n^* is an upper bound on the plant order.

THESE ASSUMPTIONS CAN NOT BE MET IN PRACTICAL SITUATIONS

INSTABILITY can result in the presence of unmodeled dynamics due to an infinite gain operator in the feedback loop.

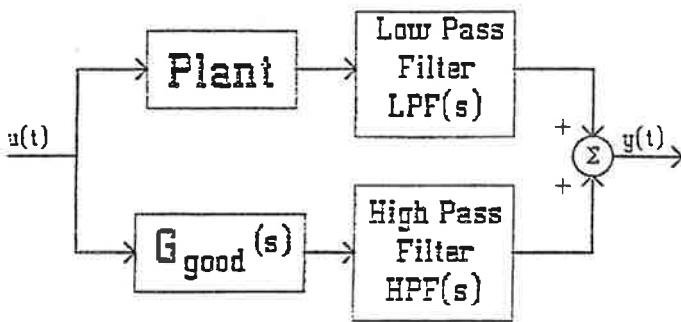


Figure 1. The conditioned plant.

At low frequencies the plant response dominates.

At high frequencies the response of G_{good} dominates.

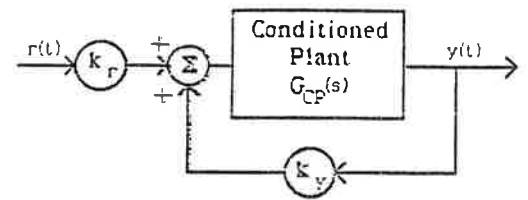
Frequency selective adaptive control is the result.

CONCLUSION

Using proper conditioning of the type described here, adaptive control can be made *robust* for any frequency reference input.

CHALLENGES

1. The inputs used here were exactly sufficiently exciting. What are the problems with under excitation and over excitation?
2. The problems with disturbances are not eliminated by conditioning.



The nominally controlled plant.

$$\lim_{s \rightarrow \infty} G_{CP}(s) = \lim_{s \rightarrow \infty} G_{good}(s) = \frac{g}{s^{n-m}}$$

g is the high frequency gain of $G_{good}(s)$ and $G_{CP}(s)$

$n-m$ is the relative degree of $G_{good}(s)$ and $G_{CP}(s)$

Assumptions 1 and 3 will be met independent of the unmodeled dynamics in the plant

If the right half plane zeroes of the plant are high frequency in nature they will not appear in G_{CP} .

Assumption 2 will be met for a large class of plants.

Simulations

Model $\frac{3}{s+3}$

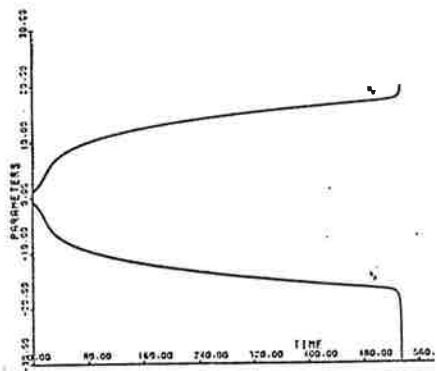
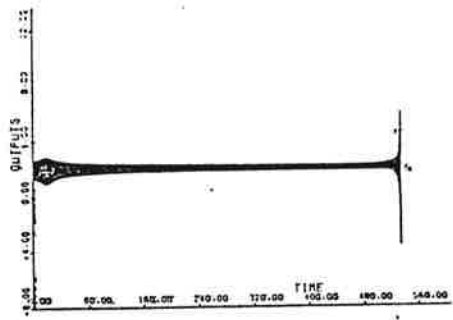
Plant $\frac{2}{s+1} \cdot \frac{229}{s^2+30s+229}$

Mechanism I

$r(x) = 2.0$ $d(x) = 0.5 \sin 16.1x$

Mechanism II

$r(x) = 2.0$ $d(x) = 0.5 \sin 8x$



0 9 0

0 0 0

L L L

Distributed asynchronous algorithms for deterministic and stochastic optimization

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There is presently a great deal of interest in distributed implementations of various iterative algorithms whereby the computational load is shared by several processors while coordination is maintained by information exchange via communication links. In most of the work done in this area the starting point is some iterative algorithm which is guaranteed to converge to the correct solution under the usual circumstances of centralized computation in a single processor. The computational load of the typical iteration is then divided in some way between the available processors, and it is assumed that the processors exchange all necessary information regarding the outcomes of the current iteration can begin.

The mode of operation described above may be termed synchronous in the sense that each processor must complete its assigned portion of an iteration and communicate the results to every other processor before a new iteration can begin. This assumption certainly enhances the orderly operation of the algorithm and greatly simplifies the convergence analysis. On the other hand synchronous distributed algorithms also have some obvious implementation disadvantages such as the need for an algorithm initiation and iteration synchronization protocol. Furthermore the speed of computation is limited to that of the slowest processor. It is thus interesting to consider algorithms that can tolerate a more flexible ordering of computation and communication between processors. Such algorithms have so far found applications in computer communication networks, e.g., ARPANET and other networks designed like it where processor failures are common and it is quite complicated to maintain synchronization between the nodes of the entire network as they execute real-time network functions such as the routing algorithm.

Processor network environments for which weakly coordinated distributed computation seems particularly advantageous typically possess one or more of the following characteristics all of which involve occurrence of some type of unpredictable event.

- 1/ Computation nodes and communication links are subject to frequent and/or unexpected failures. (For example packet radio networks).
- 2/ Computation nodes have different and/or time varying speeds of execution. (For example each processor is assigned to a perhaps time varying number of tasks involving computation loads which are not fixed a priori).
- 3/ Computations at various nodes is event driven. (For example in data collection or sensor networks where the timing, and ordering of measurements may not be predictable.).

It is possible to consider various degrees of coordination in different types of distributed algorithms. An interesting question is to determine the minimum degree of coordination needed in a given algorithm in order to obtain the correct solution. To this end we consider an extreme model of asynchronous distributed algorithms where by computation and communication are performed at each processor completely independently of the progress in the other processors. It is perhaps surprising that even under these chaotic circumstances it is still possible to solve correctly important classes of problems. An account of progress made in this direction is given in a survey jointly written with J. Tsitsiklis and M. Athans (1983). An analysis is given in (Bertsekas, 1982) for broad classes of dynamic programming problems and in (Bertsekas, 1983) for more general fixed point problems involving contractions and monotonicity assumptions. Further related work is (Tsitsiklis, Bertsekas, and Athans, 1983), and (Tsitsiklis, 1983).

References

- Bertsekas, D.P. (1982): Distributed Dynamic Programming. IEEE Trans. on Automatic Control, AC-27, 610-616.
- Bertsekas, D.P. (1983): Distributed Computation of Fixed Points. Math. Programming, 27, 107-120.
- Bertsekas, D.P., Tsitsiklis, J.N., and Athans, M. (1983): Convergence Theories of Distributed Iterative Processes: A Survey. Large Scale Systems: Theory and Applications (to appear); also LIDS Report LIDS-P-1342, Mass. Institute of Technology.
- Tsitsiklis, J.N. (1983): Problems in Distributed Decision Making. Ph.D. Thesis, Dept. of Elec. Engineering and Computer Science, Mass. Institute of Technology, May 1983.
- Tsitsiklis, J.N., Bertsekas, D.P., and Athans, M. (1983). Distributed Deterministic and Stochastic Optimization Algorithms with Applications in System Identification. Proc. of Melecon'83, Athens Greece, May 1983, paper C5.03.

Expert control

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There has been substantial progress in theory and practice of automatic control through application of mathematical analysis and numerics. Nonnumerical data processing has, however, so far only had marginal influence on control systems. The purpose of this paper is to identify possible uses of expert system techniques in implementation of control systems. It is first observed that actual implementation of control laws often involves a substantial amount of heuristic logic. This is true for simple regulators as well as for more sophisticated multivariable control loops. The paper shows that the heuristic logic may be replaced by an expert system. This leads to simplifications in implementation as well as new capabilities in the control system. Selected basic elements of an expert system are presented. Stochastic dynamic programming offers a framework in which the heuristics can be embedded. This points to requirements for a new artificial intelligence approach for heuristic planning under uncertainty. The ideas are illustrated by examples: a smart PID regulator, a self-tuner with safety jackets and a pole-placement adaptive regulator which can by itself determine suitable pole locations. Once the expert system approach is taken it is possible to obtain control systems with new functions. This is illustrated by the smart PID regulator which incorporates learning.

References

- Årzén, K.E. (1985). Experiments with expert control. To appear.
- Åström, K.J. (1982). Ziegler-Nichols Auto-tuners. Report TFRT-3167. Department of Automatic Control, Lund Institute of Technology.
- Åström, K.J. (1983). Theory and applications of adaptive control. Automatica 19, 471-486.
- Åström, K.J. and T. Hägglund (1984). Automatic tuning of simple regulators with specifications on phase and amplitude margins. Automatica 20, 645-651.
- Åström, K.J. and J. Anton (1984). Expert control. Proc. 10th IFAC World Congress Budapest.
- Barr, A. and E.A. Feigenbaum (Eds.) (1982). The Handbook of Artificial Intelligence, Volume 2. William Kaufmann, Inc., Los Altos, California.
- Davis, R. (1982). Expert systems. Where are we? And where do we go from here?. The AI Magazine, Spring 1982, 3-22.
- Hayes-Roth, F., D. Waterman and D. Lenat (1983). Building expert systems. Addison-Wesley, Reading, Mass.
- Wittenmark, B. and K.J. Åström (1984). Practical issues in the implementation of adaptive control. Automatica 20, 595-605.

EXPERT CONTROL

- [> 1. INTRODUCTION
2. CONTROL PRACTICE
3. EXPERT SYSTEMS
4. EXPERIMENTS
5. CONCLUSIONS

MOTIVATION

SAFE OPERATION OF STR
COMPUTER POWER
SYMBOLIC DATA PROCESSING

PID CONTROL

manual automatic
parameter changes
switching transients
integral windup

CONCLUSION I

Even simple regulators
combines algorithms and
heuristics

ADAPTIVE CONTROL

LOCAL GRADIENT ALGORITHMS
PRIOR KNOWLEDGE
INITIALIZATION
SAFE-GUARDS

CONCLUSION II

Adaptive controls
contain a lot of
heuristics

EXPERT SYSTEM (ES)

Data base
Facts, Evidence,
Hypotheses, Goals
Supervisory strategy

RULE BASED ES

If < > then < >
Forward chaining
Backward chaining
Why

SUCCESS STORIES

Experts and data available
Problem Scope
Combinatorics
Incremental progress

$$Ay_t = Bu_{t-d} + Ce_t$$

$$Ru = -Sy$$

$$z^{d-1}CB = AR + BS$$

$$y_t = f_0 e_t + \dots + f_{d-1} e_{t-d+1}$$

$$F = R/B$$

PRIOR KNOWLEDGE

delay
sampling period
regulator complexity
forgetting factor
initial estimates
bounds on control

OPERATOR CLASSES

Main Monitor
Backup Control
Minimum variance control
Estimation
Tuning
Learning

Main monitor:
stability-supervisor
control-quality-supervisor

Back-up control:
pid-control
auto-tune

Fixed gain MV control:
minimum-variance-control
minimum-variance-supervisor
ringing-detector
degreesupervisor

Estimation:
parameter-estimation
excitation-supervisor
perturbation-signal-generation
jump-detector

Self-tuning:
self-tuning-regulation

Learning:
get-regulator-parameters
put-regulator-parameters
store-regulator-parameters
test-scheduling-hypothesis
smooth-table-entries

Main monitoring table

#	Time	u	σ_u	y	σ_y	Stable	Reg. type

Backup control Table.

#	Time	k_c	t_c	P	I	D

Minimum variance control table

#	Time	n_R	n_d	h	Parameters

WHY USE ES?

Simple coding
Separate algorithms and logic

ALGORITHM ORCHESTRATION

Control Algorithms
Diagnosis Algorithms
Logic and sequencing
Tables for learning

IMPLEMENTATION

Vax 11/780
ES in Lisp
Algorithms in Pascal
Concurrency

EXPERIMENTS

-
- o SMART PID
 - o INTELLIGENT STR
 - o AUTOMATIC ω_B CHOICE

JUDGEMENT

Ideas probably much more important for large complex systems. But let us do simple things first.

CONCLUSIONS

-
- o New control laws
 - o Many algorithms
 - o Control and diagnosis
 - o Learning

Experiments with expert control

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Abstract:

Expert control refers to a control system where an expert system is used to orchestrate a collection of control and identification algorithms. This is done in real time. Adaptive control algorithms need large safety jackets of logic to work in practice. Existing adaptive algorithms perform well locally but require a priori information about time delay, system order etc. to do so. Simple algorithms exist that can provide some of this information. An expert system is well suited for implementation of logic. Heuristics and rules of thumb are also easily implemented.

A testbench for experiments is presented. The controller is divided in two parts. One algorithm library written in Pascal and one expert system written in OPS4 and Lisp. These parts are implemented as communicating concurrent processes. The communication is done with mailboxes and messages. A typical message is to start or stop an algorithm. OPS4 is a rule based, forward chaining expert system framework.

An experiment is presented where the level of a water tank is controlled by a PID controller with Ziegler-Nichols auto tuning and gain scheduling.

Reference:

Åström K. J. and Anton J. J.: Expert Control, Proceedings IFAC 9th World Congress, Budapest, Hungary. 1984.

EXPERIMENTS WITH EXPERT CONTROL.

- Introduction
- Implementation
- Experiments
- Demonstration

MOTIVATION

- "Safety Jacket"
- Many algorithms
- Heuristics & rules of thumb
- Make full use of a priori information
- Division: Logic ↔ Algorithm
- Workbench

Programming Languages

Symbolic Data Processing,
AI-tradition ⇒ Lisp
Prolog

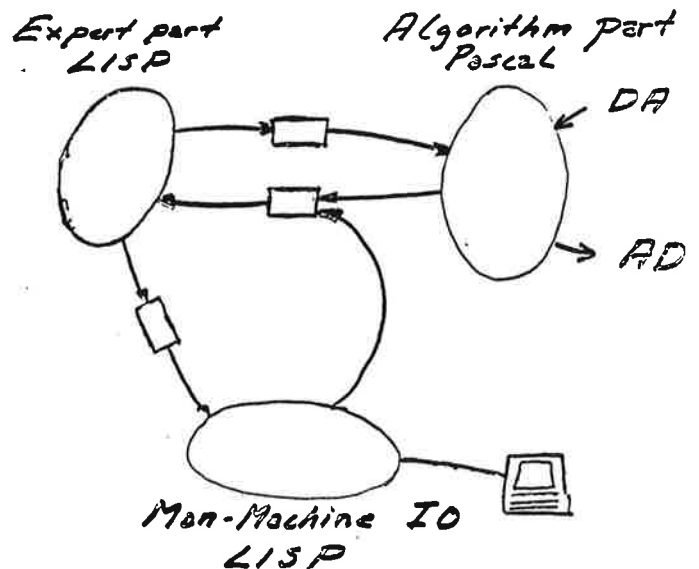
Numerical algorithms
⇒ Pascal
Fortran
ADA

Concurrency

Expert System in real time.

IMPLEMENTATION

- VAX 11/780, VMS
- Pascal, Franz Lisp



Algorithm part

- Library of control, identification and supervision algorithms with a few well specified operations upon.

• Control loop

```
while true do
  begin
    if mail-in-box then readmail;
    for all algorithms do
      if algorithm-is-active then
        execute-algorithm;
    wait;
  end;
```

EXPERIMENTS

- 1) "Smart" PID that can decide whether
 - The process can be controlled by PI
 - - " - PID
 - The process needs a more complex controller
- 2) STR that determines the prediction horizon automatically

Expert System Part

- Existing Expert System Framework

OPS 4

Consists of

Working Memory
Production - II -
Control Structure

Forward Chaining

Rule format:

Condition ...
--> Action ...

Recognize-act cycle

DEMONSTRATION

PID with Ziegler-Nichols
autotuning and gain scheduling

Algorithms

PID
Relay
Relay guard
Noisestimator

≈ 50-60 rules.

Notes from the discussion

A short summary of the discussion on Wednesday afternoon is given here. The conclusions given below should not be taken as declarations everyone agreed on, but rather as a couple of interesting statements brought up during the discussion. Some statements were accepted by most participants, other maybe by very few.

I. Process control versus aircraft control

Process control and aircraft control are two totally different issues. In aircraft control, lots of time, money and work are spent to obtain a very high performance. In process control, the control work must often be both fast and cheap, and high performance is often not so important. Stability is often enough. Furthermore, these systems have different types of dynamics. These are some differences which has influenced the success of adaptive control in the process industry, and the luck of success in the aircraft control.

II. Parametrization

Current parametrizations used in adaptive control are not good - we are probably using them only because we know how to solve the identification problem for these parametrizations.

Approximations are best done on an input-output basis, in the frequency domain. Unstructured inaccuracy would e.g. be given as nominal phase and amplitude curves \pm ranges. State-space parametrization is not good for approximation. Therefore, the state-space realization part should be left to the end of the design procedure.

State-space is excellent for rigid body dynamics.

You cannot have an adaptive theory unless you have a feedback theory that deals with plant uncertainty. Feedback reduces tolerance bound.

III. Is identification essential to adaptive control?

Identification is essential to adaptive systems. You have to obtain knowledge of the system from measurements.

From a theoretical point of view, identifiability may not be essential, but for robustness it may. Otherwise we do not catch the whole dynamics. As long as you get the I/O-map, this is enough. The number of parameters may lead to numerical problems. Even if the I/O map is not changing, the internal parameters may change. This is a numerical problem.

IV. How to take care of apriori knowledge

Prefiltering may be one way to include apriori knowledge to decrease the number of parameters.

We would like to remove as many critical parameters, such as nonminimum phase and time delay, as possible. The relay method can solve the time delay problem, at least for systems with monotone step responses.

A lot of tricks are currently added to take care of non-typical situations. It should be useful to get insight from theory why we need those tricks. Some problems are related to robustness.