

Simultaneous detection of the nonlinear restoring and excitation of a forced nonlinear oscillation: An integral approach

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Abstract

We address in this article, how to calculate the restoring characteristic and the excitation of a nonlinear forced oscillating system. Under the assumption that the forced nonlinear oscillator has a periodic solution with period $T = 2\pi/\omega$, we constructed a system of linear equations by introducing time-dependent multipliers. The periodicity assumption helps to simplify the system of linear equations. The stability and uniqueness are also presented for the inverse problem. Numerical testing is conducted to show the effectiveness of our presented methodology.

Keywords: Forced nonlinear oscillator, Inverse problem, Periodic solution, Nonlinear restoring and excitation

1. Introduction

Known as the art and science, system identification is of building mathematical models of physical systems from observed data, which has been of great significance in mathematical sciences as well as applied engineering. Especially in engineering applications, the issue of identifying a forced nonlinear dynamic system, being involved in the present work, appears to be fascinating and significant, because its understanding is of great importance for the practical design of nonlinear systems in real world. From application viewpoint, the nonlinear characteristics of a ship motion can be modeled using Duffing equation and hence their identification from experimentally measured data is an interesting area of study. Even though, there have been a lot of valuable works regarding the identification of forced systems, most studies of them are limited to only the force identification that recovers external single force excitations. For example, Doyle [1] proposed a wavelet-based deconvolution method for recovering impact forces of a system. From measurements, a method of identification of system parameters and sinusoidal and impulsive forces was suggested by Lu and Law [2]. A new mathematical procedure was proposed for measuring non-harmonic periodic excitation forces in nonlinear damped systems(Jang et al. [3]).

However, in contrast to the previous (single) force identification, this paper concerns a simultaneous identification, recovering not only the force excitations but the restoring characteristic of a forced nonlinear system. In fact, little studies have been tried for the identification of both the force excitations and the restoring characteristic in forced nonlinear systems: that is, most of progress has been made on the (single) identification of either or force excitations or restoring characteristic.

In a recent article [4], Jang proposed a mathematical process for recovering the external excitation, as well as the linear and nonlinear restoring force coefficients of a forced nonlinear oscillator. He, based on introducing J-function and its zero-crossing, used the energy balance idea to estimate the value of external excitation under assumption that nonlinear oscillator has a periodic solution. The detection of time-varying nonlinear damping in nonlinear oscillation systems is explored in [5] with the help of integral equation formulation. The idea of zero-crossing is again presented in [6] to recover amplitude and nonlinear restoring coefficients. In [7], the simultaneous detection of nonlinear damping,

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phase shift and amplitude of external harmonic excitation is studied. The designed problem was ill-posed, and a regularization scheme is used to get faithful estimation of parameters. The nonlinear oscillators are well studied by many researchers for instant see [8–16] and references therein. A valuable investigation has been conducted to detect the parameters from noisy data for forced nonlinear oscillators[17–24] and provides base for the new exploration in area of inverse problems.

In the present study we extend the idea which was presented in [4] for the detection of amplitude by introducing a set of continuous functions ψ_j . In this paper, based on the idea of [4], we would like to present a new methodology with an integral approach. We construct a system of linear equations by introducing a set of continuous multipliers. Thereby, a general procedure will be provided for calculating the two restoring coefficients and the external force. Finally, the uniqueness and the stability are also investigated and numerical experiments are given, showing that our presented methodology is efficient and accurate.

2. Inverse problem for forced nonlinear oscillator

The nonlinear forced oscillator, under the action of an external harmonic load $\Gamma \cos(\omega t)$ with amplitude $\Gamma > 0$ and frequency $\omega > 0$, can be written as

$$m\ddot{q} + \phi(\dot{q}) + k_1 q + k_2 q^3 = \Gamma \cos(\omega t), \quad (1)$$

where the constant $m > 0$ stands for mass of a particle of an oscillator and $\phi(\dot{q})$ is the nonlinear damping force. The linear and nonlinear restoring constants are denoted by k_1 and k_2 respectively in (1). We make assumption that oscillating system exhibits periodic solution with period $T (= 2\pi/\omega)$ i.e.

$$q(t) = q(t + T), \quad \text{as } t \rightarrow \infty. \quad (2)$$

In practice, we have measurements for $q(t)$ and $\dot{q}(t)$ with some noise and the goal is to restore k_1 , k_2 and Γ .

3. Formulation of inverse problem

The Eq. (1) can be written as

$$qk_1 + q^3k_2 - \cos(\omega t)\Gamma = -m\ddot{q} - \phi(\dot{q}). \quad (3)$$

Now we consider functions $\psi_i \equiv \psi_i(t, q, \dot{q})$ for $i \in \{1, 2, 3\}$ which are continuous with respect to all its arguments. We construct system of three linear equations by using (3) and ψ_i as follows

$$\begin{bmatrix} \int_{t_0}^{T+t_0} q(t)\psi_1(t, q, \dot{q})dt & \int_{t_0}^{T+t_0} q(t)^3\psi_1(t, q, \dot{q})dt & - \int_{t_0}^{T+t_0} \cos(\omega t)\psi_1(t, q, \dot{q})dt \\ \int_{t_0}^{T+t_0} q(t)\psi_2(t, q, \dot{q})dt & \int_{t_0}^{T+t_0} q(t)^3\psi_2(t, q, \dot{q})dt & - \int_{t_0}^{T+t_0} \cos(\omega t)\psi_2(t, q, \dot{q})dt \\ \int_{t_0}^{T+t_0} q(t)\psi_3(t, q, \dot{q})dt & \int_{t_0}^{T+t_0} q(t)^3\psi_3(t, q, \dot{q})dt & - \int_{t_0}^{T+t_0} \cos(\omega t)\psi_3(t, q, \dot{q})dt \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ \Gamma \end{bmatrix} = - \begin{bmatrix} \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q}))\psi_1(t, q, \dot{q})dt \\ \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q}))\psi_2(t, q, \dot{q})dt \\ \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q}))\psi_3(t, q, \dot{q})dt \end{bmatrix}, \quad (4)$$

where $t_0 \rightarrow \infty$.

4. Uniqueness of solution

We select ψ_i in a way that the determinant of the matrix on the left-hand side of Eq. 4 is non-zero. Clearly, the nonzero determinant gives us unique solution for system of linear equations for selected set of ψ_i .

5. Stability of solution

Let $\{q_n(t)\}$ be a sequence such that $\lim_{n \rightarrow \infty} q_n(t) = q(t)$ and

$$I_{i,1}(q, \dot{q}) = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} q(t) \psi_i(t, q, \dot{q}) dt,$$

$$I_{i,2}(q, \dot{q}) = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} q(t)^3 \psi_i(t, q, \dot{q}) dt,$$

$$I_{i,3}(q, \dot{q}) = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \cos(\omega t) \psi_i(t, q, \dot{q}) dt.$$

The continuity of integral and $\psi_i(t, q, \dot{q})$ produce the following results

$$\begin{aligned} \lim_{n \rightarrow \infty} I_{i,1}(q_n, \dot{q}_n) &= \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} q_n(t) \psi_i(t, q_n, \dot{q}_n) dt = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \lim_{n \rightarrow \infty} q_n(t) \psi_i(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt, \\ \lim_{n \rightarrow \infty} I_{i,2}(q_n, \dot{q}_n) &= \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} q_n(t)^3 \psi_i(t, q_n, \dot{q}_n) dt = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} (\lim_{n \rightarrow \infty} q_n(t))^3 \psi_i(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt, \\ \lim_{n \rightarrow \infty} I_{i,3}(q_n, \dot{q}_n) &= \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} \cos(\omega t) \psi_i(t, q_n, \dot{q}_n) dt = \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \cos(\omega t) \psi_i(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt, \end{aligned} \quad (5)$$

From Eqs. 5 we have

$$\begin{aligned} \lim_{n \rightarrow \infty} I_{i,1}(q_n, \dot{q}_n) &= I_{i,1}(\lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n), \\ \lim_{n \rightarrow \infty} I_{i,2}(q_n, \dot{q}_n) &= I_{i,2}(\lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n), \\ \lim_{n \rightarrow \infty} I_{i,3}(q_n, \dot{q}_n) &= I_{i,3}(\lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n). \end{aligned} \quad (6)$$

With similar reasoning, one can obtain

$$\begin{aligned} \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} (m \ddot{q}_n + \phi(\dot{q}_n)) \psi_1(t, q, \dot{q}) dt &= \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} (m \lim_{n \rightarrow \infty} \ddot{q}_n + \phi(\lim_{n \rightarrow \infty} \dot{q}_n)) \psi_1(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt, \\ \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} (m \ddot{q}_n + \phi(\dot{q}_n)) \psi_2(t, q, \dot{q}) dt &= \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} (m \lim_{n \rightarrow \infty} \ddot{q}_n + \phi(\lim_{n \rightarrow \infty} \dot{q}_n)) \psi_2(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt, \\ \lim_{\substack{t_0 \rightarrow \infty \\ n \rightarrow \infty}} \int_{t_0}^{T+t_0} (m \ddot{q}_n + \phi(\dot{q}_n)) \psi_3(t, q, \dot{q}) dt &= \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} (m \lim_{n \rightarrow \infty} \ddot{q}_n + \phi(\lim_{n \rightarrow \infty} \dot{q}_n)) \psi_3(t, \lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n) dt \end{aligned} \quad (7)$$

The Eq. 4 can be recast as

$$\begin{aligned} \mathbf{A}(q, \dot{q}) \mathbf{x}(q, \dot{q}) &= \mathbf{b}(q, \dot{q}), \\ \mathbf{x}(q, \dot{q}) &= \mathbf{A}(q, \dot{q})^{-1} \mathbf{b}(q, \dot{q}), \end{aligned} \quad (8)$$

where \mathbf{A} is the matrix and \mathbf{b} is the vector on the right-hand side in Eqs. 4. The parameters k_1, k_2, Γ vector is denoted by \mathbf{x} . The Eqs. 6, 7 and 8 collectively conclude that

$$\lim_{n \rightarrow \infty} \mathbf{x}(q_n, \dot{q}_n) = \mathbf{x}(\lim_{n \rightarrow \infty} q_n, \lim_{n \rightarrow \infty} \dot{q}_n),$$

which means that the solution vector continuously depends on q and \dot{q} and stability is the consequence of continuous dependence.

6. Calculation of parameters k_1, k_2, Γ

Our presented methodology is very general because the selection of continuous ψ_i have only one constrained i.e., $\det(\mathbf{A}) \neq 0$. We make the following selections for ψ_i

$$\psi_1 = q, \quad \psi_2 = q^3, \quad \psi_3 = \dot{q}. \quad (9)$$

The further implications of Eq. 2 gives us

$$\begin{aligned} \lim_{t_0 \rightarrow \infty} q^s(t_0 + T) &= \lim_{t_0 \rightarrow \infty} q^s(t_0) \Rightarrow \lim_{t_0 \rightarrow \infty} q^s(t) \Big|_{t_0}^{t_0+T} = 0, \\ \lim_{t_0 \rightarrow \infty} \dot{q}^s(t_0 + T) &= \lim_{t_0 \rightarrow \infty} \dot{q}^s(t_0) \Rightarrow \lim_{t_0 \rightarrow \infty} \dot{q}^s(t) \Big|_{t_0}^{t_0+T} = 0, \\ \lim_{t_0 \rightarrow \infty} \{ \sin(\omega t) \text{ or } \cos(\omega t) \} q^s(t) &\Big|_{t_0}^{t_0+T} = 0, \end{aligned} \quad (10)$$

where s is a positive integer. By using (10), we produced the following useful relationships

$$\begin{aligned} \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} q^s(t) \dot{q} dt &= -s \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} q^{s-1}(t) \dot{q}^2 dt, \quad \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} q^s(t) \dot{q}(t) dt = 0, \quad \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \dot{q}(t) \dot{q}(t) dt = 0, \\ \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \cos(\omega t) q^s(t) dt &= -\frac{s}{\omega} \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \sin(\omega t) q^{s-1}(t) \dot{q} dt, \quad \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt = \omega \lim_{t_0 \rightarrow \infty} \int_{t_0}^{T+t_0} \sin(\omega t) q(t) dt. \end{aligned} \quad (11)$$

The substitution of (9) in (4) gives

$$\lim_{t_0 \rightarrow \infty} \begin{bmatrix} \int_{t_0}^{T+t_0} q^2(t) dt & \int_{t_0}^{T+t_0} q^4(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q(t) dt \\ \int_{t_0}^{T+t_0} q^4(t) dt & \int_{t_0}^{T+t_0} q^6(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q^3(t) dt \\ \int_{t_0}^{T+t_0} q(t) \dot{q}(t) dt & \int_{t_0}^{T+t_0} q(t)^3 \dot{q}(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ \Gamma \end{bmatrix} = - \lim_{t_0 \rightarrow \infty} \begin{bmatrix} \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q})) q(t) dt \\ \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q})) q^3(t) dt \\ \int_{t_0}^{T+t_0} (m\ddot{q} + \phi(\dot{q})) \dot{q}(t) dt \end{bmatrix}. \quad (12)$$

The relationships in (11) further simplify (12)

$$\lim_{t_0 \rightarrow \infty} \begin{bmatrix} \int_{t_0}^{T+t_0} q^2(t) dt & \int_{t_0}^{T+t_0} q^4(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q(t) dt \\ \int_{t_0}^{T+t_0} q^4(t) dt & \int_{t_0}^{T+t_0} q^6(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q^3(t) dt \\ 0 & 0 & \int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ \Gamma \end{bmatrix} = \lim_{t_0 \rightarrow \infty} \begin{bmatrix} m \int_{t_0}^{T+t_0} \dot{q}^2(t) dt - \int_{t_0}^{T+t_0} \phi(\dot{q}) q(t) dt \\ 3m \int_{t_0}^{T+t_0} \dot{q}^2(t) q^2(t) dt - \int_{t_0}^{T+t_0} \phi(\dot{q}) q^3(t) dt \\ \int_{t_0}^{T+t_0} \phi(\dot{q}) \dot{q}(t) dt \end{bmatrix}. \quad (13)$$

The alternative expression for (13) is

$$\lim_{t_0 \rightarrow \infty} \begin{bmatrix} \int_{t_0}^{T+t_0} q^2(t) dt & \int_{t_0}^{T+t_0} q^4(t) dt & \frac{1}{\omega} \int_{t_0}^{T+t_0} \sin(\omega t) \dot{q}(t) dt \\ \int_{t_0}^{T+t_0} q^4(t) dt & \int_{t_0}^{T+t_0} q^6(t) dt & \frac{3}{\omega} \int_{t_0}^{T+t_0} \sin(\omega t) \dot{q}(t) q^2(t) dt \\ 0 & 0 & \int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ \Gamma \end{bmatrix} = \lim_{t_0 \rightarrow \infty} \begin{bmatrix} m \int_{t_0}^{T+t_0} \dot{q}^2(t) dt - \int_{t_0}^{T+t_0} \phi(\dot{q}) q(t) dt \\ 3m \int_{t_0}^{T+t_0} \dot{q}^2(t) q^2(t) dt - \int_{t_0}^{T+t_0} \phi(\dot{q}) q^3(t) dt \\ \int_{t_0}^{T+t_0} \phi(\dot{q}) \dot{q}(t) dt \end{bmatrix}. \quad (14)$$

The third equation in (14) provides the value of Γ

$$\Gamma = \lim_{t_0 \rightarrow \infty} \frac{\int_{t_0}^{T+t_0} \phi(\dot{q}) \dot{q}(t) dt}{\int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt} \quad \text{or} \quad \Gamma = \lim_{t_0 \rightarrow \infty} \frac{\int_{t_0}^{T+t_0} \phi(\dot{q}) \dot{q}(t) dt}{\omega \int_{t_0}^{T+t_0} \sin(\omega t) q(t) dt}. \quad (15)$$

In both values of Γ the denominator should be non-zero in (15). One of the alternative expressions for ψ_i could be $a_i q + b_i q^3 + c_i \dot{q}$. The selection of odd powers of q in the definition of ψ_i provide better numerical approximated values of parameters. The presented methodology is novel because periodicity assumption only helps to simplify the integrals and selection of ψ_i are also not very restricted. Moreover, the developed methodology can also be used to restore parameters in multi-degree of freedom systems for instance in Duffing equation with two degrees of freedom.

7. Numerical testing

In this section, we conduct two numerical tests for two different cases of the nonlinear oscillator (1). The objective is to recover the coefficients of restoring and amplitude of external excitation from noisy measurements. The perturbed data can be obtained numerically by adding white noise to the noise-free data of displacements and velocities. The noise level and percentage error are defined as

$$\delta = \frac{\|q_{\text{response}} - q^\delta\|_2}{\|q_{\text{response}}\|_2} \times 100(\%),$$

$$\text{Error}(\%) = \frac{|\text{Exact value of parameter} - \text{Numerically computed value of parameter}|}{|\text{Exact value of parameter}|} \times 100.$$

7.1. Case I

For first case we define the damping law as $\phi(\dot{q}) = c\dot{q}$. Then the first equation can be stated as

$$m\ddot{q} + c\dot{q} + k_1q + k_2q^3 = \Gamma \cos(\omega t), \quad (16)$$

and consequently (13) gets the form

$$\lim_{t_0 \rightarrow \infty} \begin{bmatrix} \int_{t_0}^{T+t_0} q^2(t) dt & \int_{t_0}^{T+t_0} q^4(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q(t) dt \\ \int_{t_0}^{T+t_0} q^4(t) dt & \int_{t_0}^{T+t_0} q^6(t) dt & - \int_{t_0}^{T+t_0} \cos(\omega t) q^3(t) dt \\ 0 & 0 & \int_{t_0}^{T+t_0} \cos(\omega t) \dot{q}(t) dt \end{bmatrix} \begin{bmatrix} k_1 \\ k_2 \\ \Gamma \end{bmatrix} = \lim_{t_0 \rightarrow \infty} \begin{bmatrix} m \int_{t_0}^{T+t_0} \dot{q}^2(t) dt \\ 3m \int_{t_0}^{T+t_0} \dot{q}^2(t) q^2(t) dt \\ \int_{t_0}^{T+t_0} c \dot{q}^2(t) dt \end{bmatrix}.$$

The Eq. (16) is integrated up to $23T$ to get the numerical data and the Figure 1 shows the profiles of displacement and velocity. The Figure 2 tells the evolution of displacement and velocity for a period T and Figure 3 depicts the phase diagrams of closed orbits as well as white-noise effected closed orbit. In order to capture the real effect of white-noise on the restoring coefficients and amplitude, we run the simulations 2000 times for each noise level. We list maximum, minimum, mean percentage errors with respect to different noise levels in the Table 1. The maximum percentage error in all parameters is approximately equal to $\text{Mean} + 5\sigma$, here σ is the standard deviation. The error in displacement and velocity profiles generated from the numerically recovered parameters (without white-noise) is plotted in Figure 4. The increase in noise level enhances the error in the recovered values of restoring coefficients as well as that of in amplitude. The percentage error in linear coefficient is higher in comparison with amplitude and nonlinear coefficient in the first case. The recovered amplitude has less standard deviation and hence gives a best estimate for actual excitation.

7.2. Case II

In the present case, we use the same damping force model but the simulations are performed in different setting of parameters. The Figures 5, 6, 7 shed light on various aspects of simulations for the second case. The numerical results in Table 2 are in good agreement with that of Table 1 in the sense that the maximum percentage errors are approximately equal to $\text{Mean} + 5\sigma$. Again the percentage error in the linear coefficient is higher compare with other parameters. The estimated value of external excitation is more accurate and stable because it has less error in a mean value as well in standard deviation. Simulation values of k_1 , k_2 and Γ are plotted in Figures 8, 9 and 10 respectively. It is clearly evident to get realistic estimates of parameters, we required suitable number of data sets of observed values of displacements and velocities.

Our analysis reveals that the presented methodology provides reasonable estimates for the restoring coefficients as well as for the amplitude. Our proposal is very general in the sense that we have freedom in the selection of continuous multipliers $\psi(t, q, \dot{q})$ for the integration of nonlinear oscillator over a single period of time to get a system of linear equations which in turns provide the good estimates of parameters. In the present study, we only explore a single choice for ψ but further investigation can be carried out for the best selection of $\psi(t, q, \dot{q})$.

8. Conclusions

The presented methodology provides a very general method for the estimation of restoring coefficients and external excitation by selecting different continuous multipliers $\psi(t, q, \dot{q})$. The periodicity assumption helps to simplify the resulting system of a linear equation with parameters as unknown quantities. The developed formalism is a well-posed problem and does not face numerical instabilities. The maximum percentage error falls under $\text{Mean} + 5\sigma$ for all parameters in the present study to recover parameters from noisy data.

Acknowledgment

This work was supported by a 2-Year Research Grant of Pusan National University.

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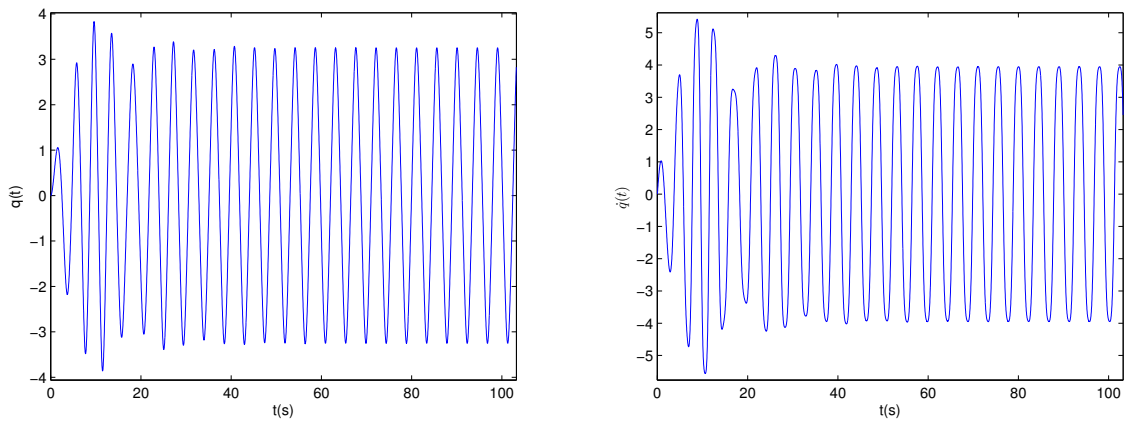


Figure 1: Case 1: displacement versus time(left), velocity versus time(right), $m = 1$, $c = 0.2$, $\omega = 1.4$, $t_0 = 21T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

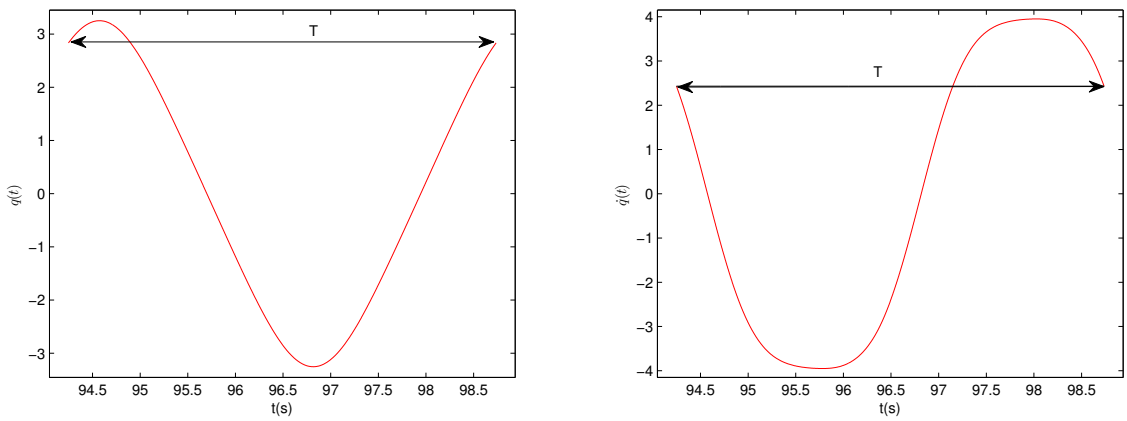


Figure 2: Case 1: displacement versus time(left), velocity versus time(right), $m = 1$, $c = 0.2$, $\omega = 1.4$, $t_0 = 21T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

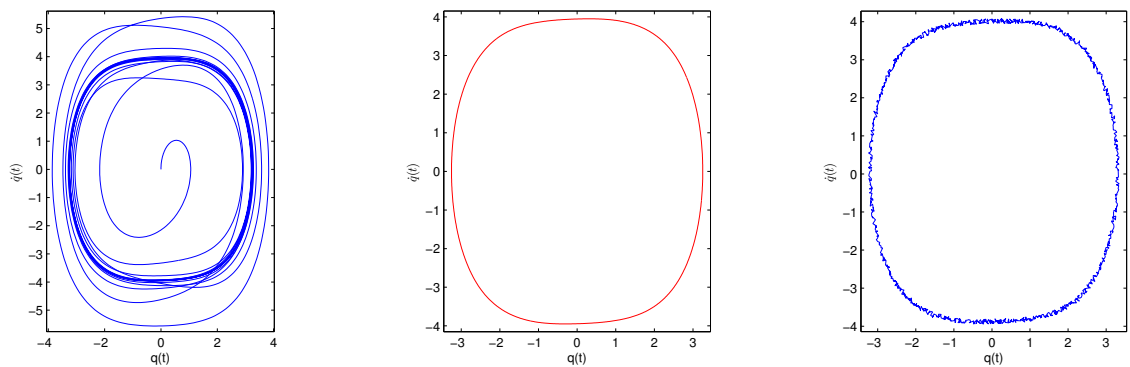


Figure 3: Case 1: orbits in phase space(left portrait convergence to closed orbits, center portrait without white noise, right portrait contaminated with 2.5% white noise), $m = 1$, $c = 0.2$, $\omega = 1.4$, $t_0 = 21T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

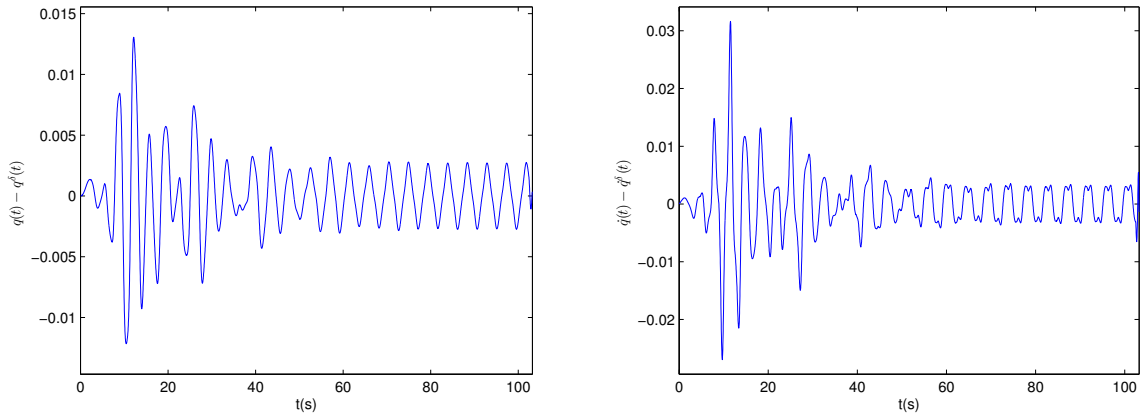


Figure 4: Case 1: Error plot of $q(t)$ and $\dot{q}(t)$ when percentage errors in Γ, k_1, k_2 are $5.4932 \times 10^{-2}, 2.1318 \times 10^{-1}, 2.9335 \times 10^{-1}$ respectively, $m = 1, c = 0.2, \omega = 1.4, t_0 = 21T, t = [0, 23T], dt = 0.02, q(0) = \dot{q}(0) = 0$

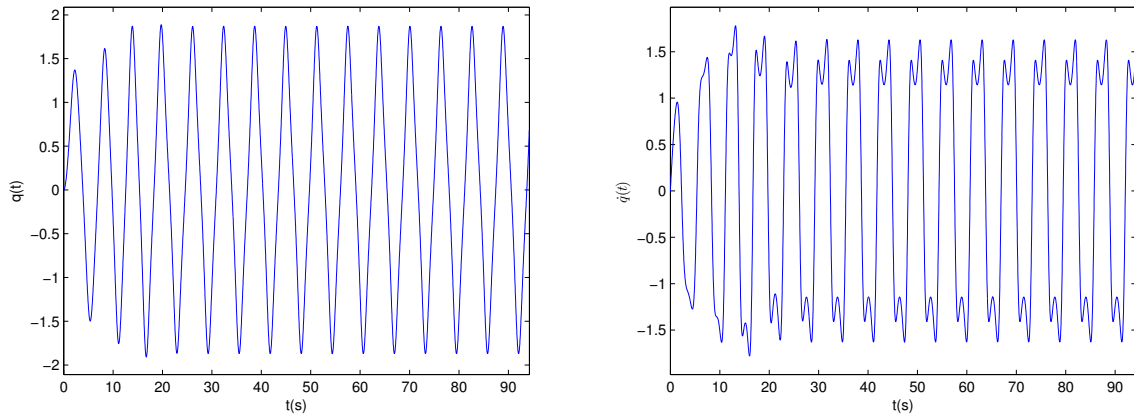


Figure 5: Case 2: displacement versus time(left), velocity versus time(right), $m = 1, c = 0.5, \omega = 1.0, t_0 = 13T, t = [0, 23T], dt = 0.02, q(0) = \dot{q}(0) = 0$

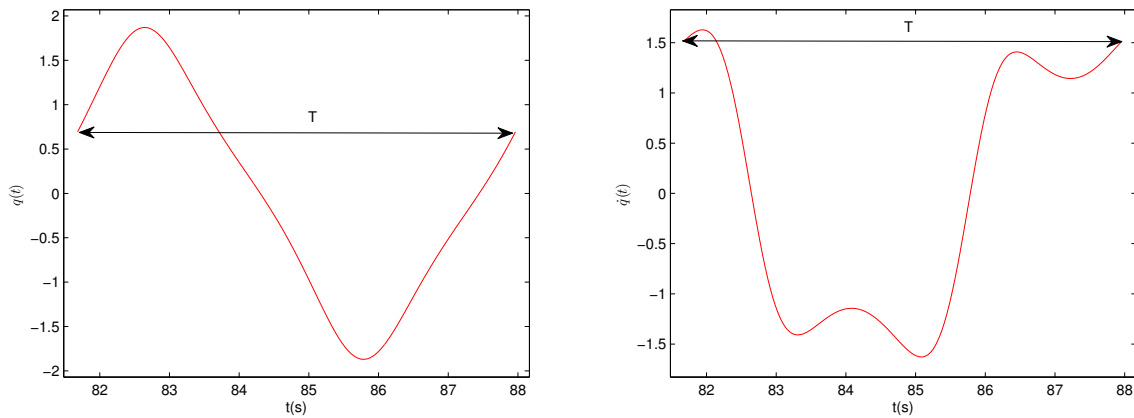


Figure 6: Case 2: displacement versus time(left), velocity versus time(right), $m = 1, c = 0.5, \omega = 1.0, t_0 = 13T, t = [0, 23T], dt = 0.02, q(0) = \dot{q}(0) = 0$

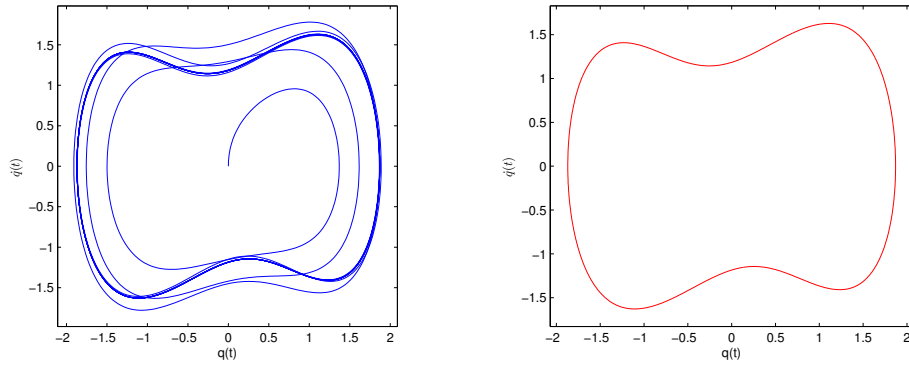


Figure 7: Case 2: orbits in phase space, $m = 1$, $c = 0.5$, $\omega = 1.0$, $t_0 = 13T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

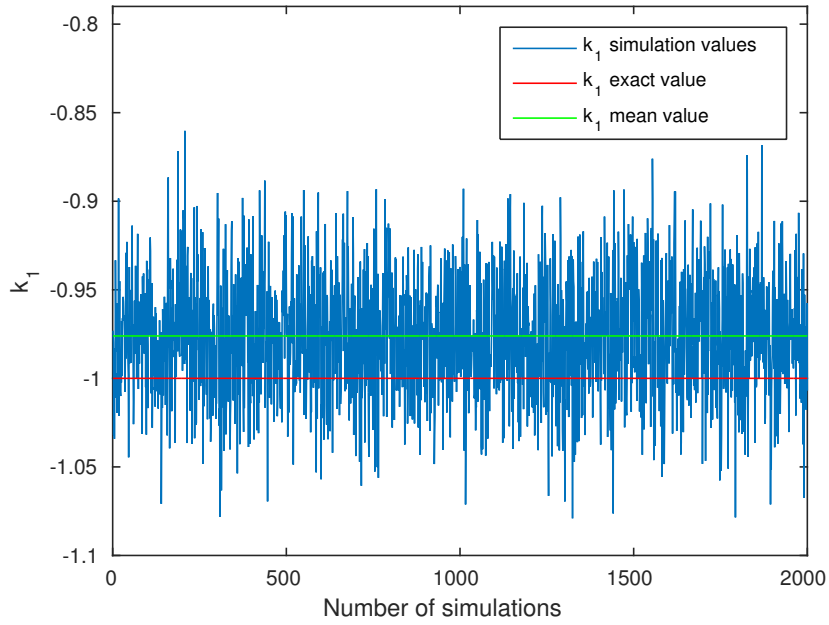


Figure 8: Case 2: k_1 simulation values at noise level 2.5%, $m = 1$, $c = 0.5$, $\omega = 1.0$, $t_0 = 13T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

Number of simulations		Γ		k_1		k_2	
2000		Amplitude		Linear coefficient		Nonlinear coefficient	
Exact		2.0		1.0		0.2	
Noise level	Numerical	Error(%)	Numerical	Error(%)	Numerical	Error(%)	
$\delta = 0$	1.9989	$5.4932e - 2$	1.0021	$2.1318e - 1$	$1.9941e - 1$	$2.9335e - 1$	
$\delta = 0.5\%$	Max	$1.7675e + 1$	Max	4.1054	Max	2.9303	
	Min	$3.8554e - 5$	Min	$8.6371e - 4$	Min	$1.8851e - 4$	
	Mean	$5.5184e - 2$	Mean	$9.1969e - 1$	Mean	0.65478	
	σ	$3.2772e - 2$	σ	$6.8277e - 1$	σ	0.48683	
	5σ	$2.1905e - 1$	5σ	4.3335	5σ	3.0889	
$\delta = 1.0\%$	Max	$2.7177e - 1$	Max	9.8332	Max	6.5622	
	Min	$7.7426e - 5$	Min	$1.8461e - 3$	Min	$4.8353e - 5$	
	Mean	$6.7037e - 2$	Mean	1.8303	Mean	1.249	
	σ	$4.7218e - 2$	σ	1.3781	σ	$9.44e - 1$	
	5σ	$3.0312e - 1$	5σ	8.7208	5σ	5.969	
$\delta = 1.5\%$	Max	$3.8822e - 1$	Max	$1.222e + 1$	Max	8.4966	
	Min	$4.4493e - 5$	Min	$5.9917e - 3$	Min	$2.4786e - 3$	
	Mean	$8.8763e - 2$	Mean	2.8246	Mean	1.9298	
	σ	$6.6917e - 2$	σ	2.059	σ	1.3922	
	5σ	$4.2335e - 1$	5σ	$1.3119e + 1$	5σ	8.8907	
$\delta = 2.0\%$	Max	$5.0268e - 1$	Max	$1.6084e + 1$	Max	$1.0613e + 1$	
	Min	$1.1417e - 3$	Min	$2.2156e - 3$	Min	$1.7996e - 1$	
	Mean	$1.1251e - 1$	Mean	3.7244	Mean	2.5463	
	σ	$8.6938e - 2$	σ	2.8492	σ	1.9284	
	5σ	$5.472e - 1$	5σ	$1.797e + 1$	5σ	$1.2188e + 1$	
$\delta = 2.5\%$	Max	$5.9335e - 1$	Max	$2.0153e + 1$	Max	$1.3395e + 1$	
	Min	$1.4682e - 4$	Min	$5.3766e - 3$	Min	$2.1279e - 3$	
	Mean	$1.418e - 1$	Mean	4.7327	Mean	3.2515	
	σ	$1.0793e - 1$	σ	3.5931	σ	2.4403	
	5σ	$6.8143e - 1$	5σ	$2.2698e + 1$	5σ	$1.5453e + 1$	

Table 1: Case 1: Errors according to the different noise levels, $m = 1$, $c = 0.2$, $\omega = 1.4$, $t_0 = 21T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

Number of simulations		Γ		k_1		k_2	
2000		Amplitude		Linear coefficient		Nonlinear coefficient	
Exact		1.1		-1.0		1.0	
Noise level	Numerical	Error(%)	Numerical	Error(%)	Numerical	Error(%)	
$\delta = 0$	1.1013	$1.1628e - 1$	$-9.9945e - 1$	$5.4864e - 2$	$9.9935e - 1$	$6.4593e - 2$	
$\delta = 0.5\%$	Max	$1.774e - 1$	Max	2.4499	Max	1.171	
	Min	$5.4243e - 2$	Min	$1.6796e - 4$	Min	$1.2237e - 5$	
	Mean	$1.1866e - 1$	Mean	$5.6558e - 1$	Mean	$2.6294e - 1$	
	σ	$1.9633e - 2$	σ	$4.0827e - 1$	σ	$1.9281e - 1$	
	5σ	$2.1683e - 1$	5σ	2.6069	5σ	1.227	
$\delta = 1.0\%$	Max	$2.5865e - 1$	Max	5.3898	Max	2.4047	
	Min	$5.8067e - 3$	Min	$1.0386e - 3$	Min	$1.2484e - 4$	
	Mean	$1.2858e - 1$	Mean	1.123	Mean	$5.1704e - 1$	
	σ	$3.9666e - 2$	σ	$8.4979e - 1$	σ	$3.8348e - 1$	
	5σ	$3.2691e - 1$	5σ	5.3719	5σ	2.4344	
$\delta = 1.5\%$	Max	$3.209e - 1$	Max	7.8892	Max	3.6193	
	Min	$9.0453e - 4$	Min	$6.2911e - 4$	Min	$9.8706e - 4$	
	Mean	$1.3792e - 1$	Mean	1.7835	Mean	$8.1135e - 1$	
	σ	$5.6846e - 2$	σ	1.3373	σ	$6.0597e - 1$	
	5σ	$4.2215e - 1$	5σ	8.4701	5σ	3.8412	
$\delta = 2.0\%$	Max	$4.12093e - 1$	Max	$1.1175e + 1$	Max	4.5748	
	Min	$5.4512e - 4$	Min	$2.0216e - 3$	Min	$2.8116e - 3$	
	Mean	$1.5525e - 1$	Mean	2.6298	Mean	1.2045	
	σ	$7.5874e - 2$	σ	1.9432	σ	$8.7542e - 1$	
	5σ	$5.3462e - 1$	5σ	$1.2346e + 1$	5σ	5.5816	
$\delta = 2.5\%$	Max	$5.7125e - 1$	Max	$1.4762e + 1$	Max	6.5401	
	Min	$1.3772e - 4$	Min	$7.2127e - 4$	Min	$2.1058e - 4$	
	Mean	$1.8006e - 1$	Mean	3.4881	Mean	1.6099	
	σ	$9.4615e - 2$	σ	2.5305	σ	1.1529	
	5σ	$6.5314e - 1$	5σ	$1.6141e + 1$	5σ	7.3746	

Table 2: Case 2: Errors according to the different noise levels, $m = 1$, $c = 0.5$, $\omega = 1.0$, $t_0 = 13T$, $t = [0, 15T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

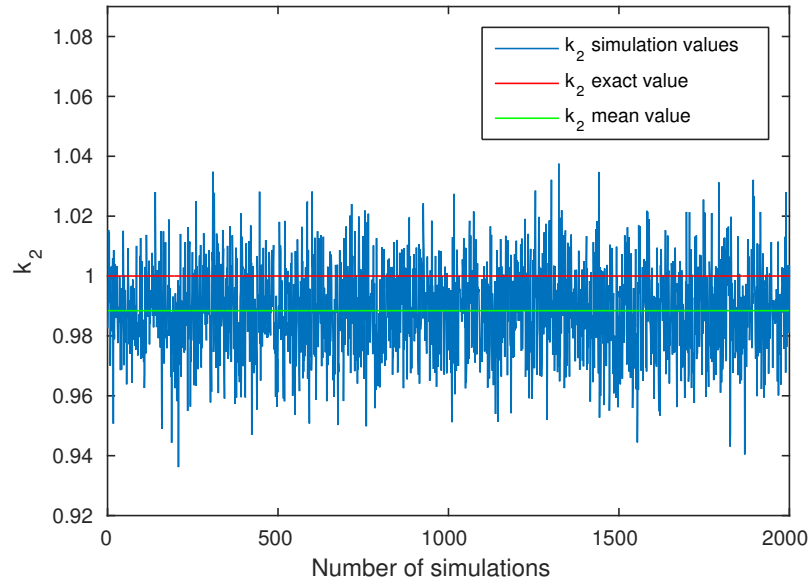


Figure 9: Case 2: k_2 simulation values at noise level 2.5%, $m = 1$, $c = 0.5$, $\omega = 1.0$, $t_0 = 13T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$

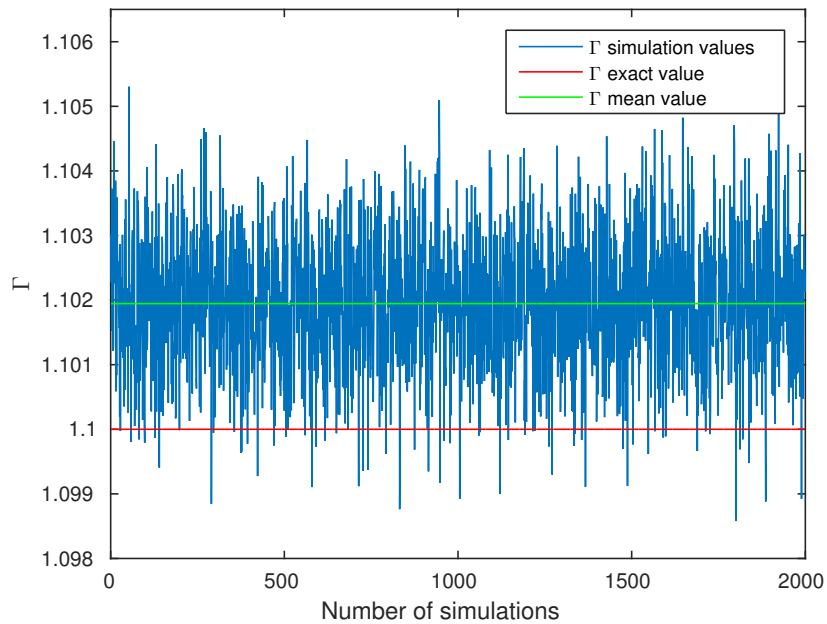


Figure 10: Case 2: Γ simulation values at noise level 2.5%, $m = 1$, $c = 0.5$, $\omega = 1.0$, $t_0 = 13T$, $t = [0, 23T]$, $dt = 0.02$, $q(0) = \dot{q}(0) = 0$